TABLE 4.3-1 (SHEET 1 OF 3)

REACTOR CORE DESCRIPTION

Active core Equivalent diameter (in.) Active fuel height (in.) Height-to-diameter ratio Total cross section area (ft²) H ₂ O/U molecular ratio, lattice, cold	LOPAR: VANTAGE 5:	132.7 143.7 1.08 96.06 2.41 2.73
Reflector thickness and composition Top - water plus steel (in.) Bottom - water plus steel (in.) Side - water plus steel (in.)		10 10 15
Fuel assemblies Number Rod array Rods per assembly Rod pitch (in.) Overall transverse dimensions (in.)		193 17 x 17 264 0.496 8.426 x 8.426
Fuel weight, as UO ₂ (lb)	LOPAR: VANTAGE 5:	222,762 204,231 ^(a)
Zircaloy weight (lb) (active core) Zircaloy/ZIRLO/Optimized ZIRLO weight (lb) (active core)	LOPAR: VANTAGE 5:	45,296 45,914
Number of grids per assembly	LOPAR: VANTAGE 5:	8 R type 2 nonmixing vane type, 6 mixing vane type, 3 IFM
Composition of grids	LOPAR: VANTAGE 5:	1 protective grid Inconel-718 2 Inconel-718 end grids 6 Zircaloy-4/ZIRLO/Low Tin ZIRLO spacer grids 3 Zircaloy-4/ZIRLO/Low Tin ZIRLO IFM grids 1 Inconel-718 protective grid
Weight of grids in active core (lb)	LOPAR: VANTAGE 5:	Inconel-718 – 2324 Inconel-718 – 332 Zircaloy-4/ZIRLO/Low Tin ZIRLO 3547
Number of guide thimbles per assembly Composition of guide thimbles		24 Zircaloy-4/ZIRLO

TABLE 4.3-1 (SHEET 2 OF 3)

Diameter of guide thimbles, upper part (in.)	LOPAR:	0.450 ID x 0.482 OD
	VANTAGE 5:	0.442 ID x 0.474 OD
Diameter of guide thimbles, lower part (in.)	LOPAR:	0.397 ID x 0.430 OD
	VANTAGE 5:	0.397 ID x 0.430 OD
Diameter of instrument guide thimbles (in.)	LOPAR:	0.450 ID x 0.482 OD
	VANTAGE 5:	0.442 ID x 0.474 OD
Fuel rods		0.5
Number	LODAD	50,952
Outside diameter (in.)	LOPAR: VANTAGE 5:	0.374 0.360
Diameter gap (in.)	LOPAR:	0.0065 0.0062
Clad thickness (in.)		0.0225
Clad material		Zircaloy-4/ZIRLO/ Optimized ZIRLO
Fuel pellets (first cycle)		Optimized Zirkeo
Material Control of the control of t		UO ₂ sintered
Density (% of theoretical) First Cycle fuel enrichments (weight percent)		95
Region 1		2.10
Region 2		2.60
Region 3		3.10
Diameter (in.)		0.3225
Length (in.)	Unit 1:	0.530
	Unit 2:	0.387
Mass of UO ₂ per ft of fuel rod (lb/ft)	Unit 1:	0.366
Fuel pellets (typical reload)	Unit 2:	0.364
Material		UO ₂ sintered
Density (% of theoretical)		95
Diameter (in.)	LOPAR:	0.3225
	VANTAGE 5:	0.3088 (non-IFBA)
Length (in.)	LOPAR:	0.387
	VANTAGE 5: Axial Blanket	0.370
	Pellet:	0.462/0.500

TABLE 4.3-1 (SHEET 3 OF 3)

Mass of UO₂ per ft of fuel rod (lb/ft) LOPAR: 0.364 VANTAGE 5: 0.334^(a)

RCCAs

Neutron absorber Hafnium or Ag-In-Cd

Diameter (in.) 0.341

Density (lb/in.3) Hafnium 0.454, Ag-In-Cd 0.367

Cladding material Type 304, cold-worked SS^(b)

Clad thickness (in.)

Number of clusters, full-length

53

Number of absorber rods per cluster

24

BA rods (first cycle)

Number 1518

Material Borosilicate glass

 $\begin{array}{lll} \text{OD (in.)} & 0.381 \\ \text{Inner tube, OD (in.)} & 0.1805 \\ \text{Clad material} & \text{SS} \\ \text{Inner tube material} & \text{SS} \\ \text{Boron loading (without B}_2\text{O}_3 \text{ in glass rod)} & 12.5 \\ \text{Weight of boron-10 per foot of rod (lb/ft)} & 0.00419 \\ \text{Initial reactivity worth (}\%\Delta_{\text{P}}\text{)} & ~7.6 \text{ (hot)} \end{array}$

al reactivity worth (% $\Delta \rho$) ~7.6 (not) ~5.5 (cold)

Burnable Absorbers (reload cycles)

Wet Annular Burnable Absorber Rods:

 $\begin{array}{lll} \text{Material} & \text{A1}_2\text{O}_3\text{-B}_4\text{C} \\ \text{OD (in.)} & \text{0.381} \\ \text{Inner tube, OD (in.)} & \text{0.267} \\ \text{Clad material} & \text{Zircaloy} \\ \text{Inner tube material} & \text{Zircaloy} \\ \text{B}_{10} \text{ content (mg/cm)} & \text{6.03} \\ \end{array}$

Integral Fuel Burnable Absorbers:

Material ZrB₂

Typical B_{10} content (mg/in.) 1.50 to 2.25 (1.0X to 1.5X)

Excess reactivity (first cycle)

Maximum fuel assembly K∞ (cold, clean, 1.39

unborated water)

Maximum core reactivity (cold, zero power, 1.222

beginning of cycle, zero soluble boron)

a. The decrease in fuel weight due to annular axial blanket pellets is not considered.

b. 316L SS cladding material applicable to Framatome RCCA only.

c. (Note that the PRIME™ Fuel Assembly design adds an external sleeve (0.442 inch ID x 0.482 inch OD) which slips over the dashpot (lower part) of the guide thimble tube and is bulged in place above the bottom grid.)

TABLE 4.3-2 (SHEET 1 OF 2)

NUCLEAR DESIGN PARAMETERS

(First Cycle)

Core average linear power, including densification effects (kW/ft)	5.45	
Total heat flux hot channel factor, $F_{\mathtt{Q}}$	2.32	
Nuclear enthalpy rise hot channel factor, $ F_{\Delta H}^{N} $	1.55	
Reactivity coefficients $^{(a)}$ Doppler-only power coefficients, see figure 15.1-5, (pcm/% power) $^{(b)}$	<u>Design Limits</u>	Best Estimate
Upper curve Lower curve Doppler temperature coefficient $(\text{pcm}/^{\circ}\text{F})^{(b)}$ Moderator temperature coefficient $(\text{pcm}/^{\circ}\text{F})^{(b)}$ Boron coefficient $(\text{pcm/pm})^{(b)}$ Rodded moderator density $(\text{pcm/g/cm})^{3(b)}$	-19.4 to $-12.6-10.2$ to $-6.7-2.9$ to $-1.40 to -40-12.8$ to $-7.5\le 0.43 \times 10^5$	-15 to -11 -13 to -9 -2.4 to -1.7 -1 to -36 -16 to -7 ≤0.35 x 10 ⁵
Delayed neutron fraction and lifetime		
β_{eff} BOL, (EOL)	0.0075	
€*, BOL, (EOL) μs	(1100.0)	19.4 (18.1)
Control rods Rod requirements Maximum bentk worth (pcm) Maximum ejected rod worth Bank worth HZP no overlap (pcm) ^(b)	See table 4.3-3. <2000 See chapter 15. BOL, Xe free	EOL Eq. Xe
Bank D Bank C Bank B Bank A	650 1250 1200 500	750 1450 1400 450
Radial factor (BOL to EOL) Unrodded D bank D + C banks D + C + B banks	1.37 to 1.28 1.50 to 1.45 1.60 to 1.45 1.80 to 1.55	

TABLE 4.3-2 (SHEET 2 OF 2)

	1435	1408	2000	1327	1307	1178	882		See figure 4.3-3.	~100
Boron concentrations (ppm)	Zero power, $k_{eff} = 0.99$, cold ⁽²⁾ RCCAs out	Zero power, $k_{eff} = 0.99$, hot ^(e) RCCAs out	Design basis refueling boron concentration	Zero power, k _{eff} ≤ 0.95, cold ^(d) RCCAs in	Zero power, k _{eff} = 1.00, hot ^(e) RCCAs out	Full power, no xenon, $k_{eff} = 1.0$, hot RCCAs out	Full power, equilibrium xenon, $k_{eff} = 1.0$, hot RCCAs out	Reduction with fuel burnup	First cycle (ppm/GWd/tonne uranium) ^(f)	Reload cycle (ppm/GWd/tonne uranium)

a. Uncertainties are given in paragraph 4.3.3.3.

b. 1 pcm = $10^{-5} \Delta \rho$ where $\Delta \rho$ is calculated from two statepoint values of k_{eff} by 1n (k_1/k_2).

c. Bounding lower value used for safety analysis.

d. Cold means 68°F, 1 atm.

e. Hot means 557°F, 2250 psia.

f. 1 GWd = 1000 MWd. During the first cycle, fixed BP rods are present which significantly reduce the boron depletion rate compared to reload cycles.

VEGP-FSAR-4

TABLE 4.3-3

REACTIVITY REQUIREMENTS FOR ROD CLUSTER CONTROL ASSEMBLIES

Rea	Reactivity Effects (Percent)	BOL (First Cycle)	EOL (First Cycle)	EOL Representative Equilibrium Cycle)
-	Control requirements Fuel temperature, Doppler (% $\Delta \rho$) Moderator temperature (% $\Delta \rho$) Redistribution (% $\Delta \rho$) Rod insertion allowance (% $\Delta \rho$)	1.37 0.15 0.50 0.50	1.21 1.15 0.85 0.50	1.10 1.15 0.98 0.50
2	Total control (%Др)	2.52	3.71	3.73
က်	Estimated RCCA worth (53 rods) a. All full-length assemblies inserted (% $\Delta \rho$)	7.54	7.42	6.76
4.	b. All assemblies but one (highest worth) inserted $(\%\Delta\rho)$ Estimated RCCA credit with 10-percent adjustment to accommodate uncertainties, item 3b minus 10 percent $(\%\Delta\rho)$	6.46 5.82	6.39 5.75	5.78 5.20
Ŋ.	Shutdown margin available, item 4 minus item 2 (% $\Delta \rho$)	3.30	2.04	1.47 ^(b)

a. Includes void effects.

b. The design basis minimum shutdown is 1.3 percent.

TABLE 4.3-4

DELETED

TABLE 4.3-5

AXIAL STABILITY INDEX PRESSURIZED WATER REACTOR CORE WITH A 12-FT HEIGHT

Burnup (MWd/tonne <u>uranium)</u>	<u>F_Z</u>	C _B (ppm)	Stability Ind	dex (h ⁻¹) Calc
1550	1.34	1065	-0.041	-0.032
7700	1.27	700	-0.014	-0.006
5090 ^(a)			-0.0325	-0.0255
2250 ^(b)		Radial St	ability Index -0.068	-0.07

a. Four-loop plant, 12-ft core in cycle 1, axial stability test.

b. Four-loop plant, 12-ft core in cycle 1, radial (X-Y) stability test.

	E>1.0 MeV	0.111 MeV < E <1.0 MeV	0.3 eV ≤ E <0.111 MeV	<e 0.3="" ev<="" th=""></e>
Core center	9.98 x 10 ¹³	1.11 x 10 ¹⁴	2.17 x 10 ¹⁴	5.36 x 10 ¹³
Core outer radius at midheight	4.24 x 10 ¹³	4.85 x 10 ¹³	9.52 x 10 ¹³	2.21 x 10 ¹³
Core top, on axis	2.62 x 10 ¹³	2.13 x 10 ¹³	1.31 x 10 ¹⁴	4.35 x 10 ¹³
Core bottom, on axis	2.70 x 10 ¹³	2.25 x 10 ¹³	1.33 x 10 ¹⁴	4.74 x 10 ¹³
Pressure vessel ID azimuthal peak,	2.08 x 10 ¹⁰	2.83 x 10 ¹⁰	6.18 x 10 ¹⁰	1.20 x 10 ¹¹

TABLE 4.3-7
COMPARISON OF MEASURED AND CALCULATED DOPPLER DEFECTS

<u>Plant</u>	Fuel Type	Core Burnup (MWd/tonne <u>uranium)</u>	Measured (pcm) ^(a)	Calculated (pcm)
1	Air filled	1800	1700	1710
2	Air filled	7700	1300	1440
3	Air and helium filled	8460	1200	1210

a. pcm = 10^5 x ln (k/k)

TABLE 4.3-8
SAXTON CORE II ISOTOPICS ROD MY, AXIAL ZONE 6

Atom Ratio	Measured ^(a)	2 σ Precision (%)	LEOPARD Calculation
U-234/U U-235/U U-236/U U-238/U	4.65 x 10 ⁻⁵ 5.74 x 10 ⁻³ 3.55 x 10 ⁻⁴ 0.99386	±29 ±0.9 ±5.6 ±0.01	4.60 x 10 ⁻⁵ 5.73 x 10 ⁻³ 3.74 x 10 ⁻⁴ 0.99385
Pu-238/Pu Pu-239/Pu Pu-240/Pu Pu-241/Pu Pu-242/Pu	1.32 x 10 ⁻³ 0.73791 0.19302 6.014 x 10 ⁻² 5.81 x 10 ⁻³	±2.3 ±0.03 ±0.2 ±0.3 ±0.9	1.222 x 10 ⁻³ 0.74497 0.19102 5.74 x 10 ⁻² 5.38 x 10 ⁻³
Pu/U ^(b)	5.938 x 10 ⁻²	±0.7	5.970 x 10 ⁻²
Np-237/U-238	1.14 x 10 ⁻⁴	±15	0.86 x 10 ⁻⁴
Am-241/Pu-239	1.23 x 10 ⁻²	±15	1.08 x 10 ⁻²
Cm-242/Pu-239 Cm-244/Pu-239	1.05 x 10 ⁻⁴ 1.09 x 10 ⁻⁴	±10 ±20	1.11 x 10 ⁻⁴ 0.98 x 10 ⁻⁴

a. Reported in reference 34.

b. Weight ratio.

TABLE 4.3-9
CRITICAL BORON CONCENTRATIONS (ppm) HZP, BOL

Plant Type	<u>Measured</u>	Calculated
2-loop, 121 assemblies, 10-ft core	1583	1589
2-loop, 121 assemblies, 12-ft core	1625	1624
2-loop, 121 assemblies, 12-ft core	1517	1517
3-loop, 157 assemblies, 12-ft core	1169	1161
3-loop, 157 assemblies, 12-ft core	1344	1319
4-loop, 193 assemblies, 12-ft core	1370	1355
4-loop, 193 assemblies, 12-ft core	1321	1306

TABLE 4.3-10

COMPARISON OF MEASURED AND CALCULATED AG-IN-CD ROD WORTH

2-Loop Plant, 121 Assemblies, 10-ft Core	Measured (pcm)	Calculated (pcm)
Group B	1885	1893
Group A	1530	1649
Shutdown group	3050	2917
ESADA critical, 0.69-in. pitch ^(a) 2 8% Pu-240, 9 control rods	ł w/o PuO ₂ ,	
6.21-in. rod separation	2250	2250
2.07-in. rod separation	4220	4160
1.38-in. rod separation	4100	4019

BENCHMARK CRITICAL EXPERIMENT HAFNIUM CONTROL ROD WORTH

Control	No. of	Measured ^(b)	Calculated ^(b)
Rod	Fuel	Worth	Worth
Configuration	Rods	(∆ppm B-10)	(∆ppm B-10)
9 hafnium rods	1192	138.3	141.0

a. Reported in reference 35.

b. Calculated and measured worths are given in terms of an equivalent charge in B-10 concentration.

TABLE 4.3-11

COMPARISON OF MEASURED AND CALCULATED MODERATOR

COEFFICIENTS AT HZP, BOL

Plant Type/ Control Bank Configuration	Measured α _{iso} ^(a) <u>(pcm/°F)</u>	Calculated α _{iso} (pcm/°F)
3-loop, 157-assembly, 12-ft core		
D at 160 steps D in, C at 190 steps D in, C at 28 steps B, C, and D in	-0.50 -3.01 -7.67 -5.16	-0.50 -2.75 -7.02 -4.45
2-loop, 121-assembly, 12-ft core D at 180 steps D in, C at 180 steps C and D in, B at 165 steps B, C, and D in, A at 174 steps	+0.85 -2.40 -4.40 -8.70	+1.02 -1.90 -5.58 -8.12
4-loop, 193-assembly, 12-ft core		
All Rods Out D in D and C in D, C, and B in D, C, B, and A in	-0.52 -4.35 -8.59 -10.14 -14.63	-1.2 -5.7 -10.0 -10.55 -14.45

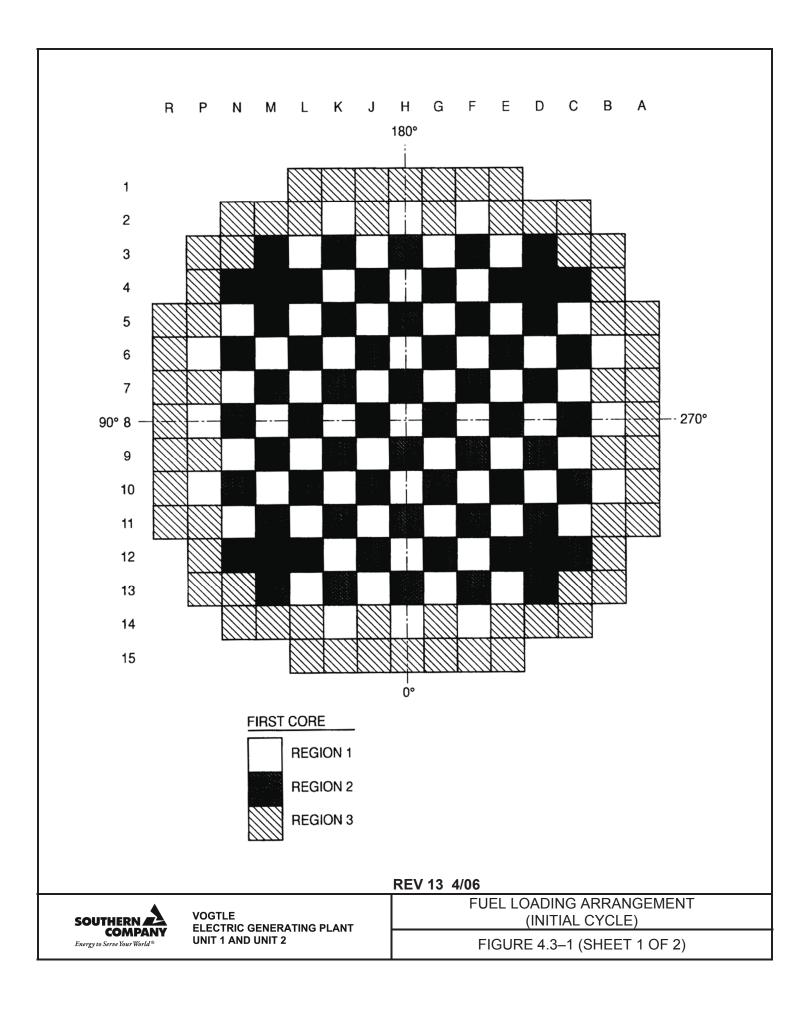
$$\alpha_{iso} = \frac{10^5 \ln \frac{k_2}{k_1}}{\Delta T^{\circ} F}$$

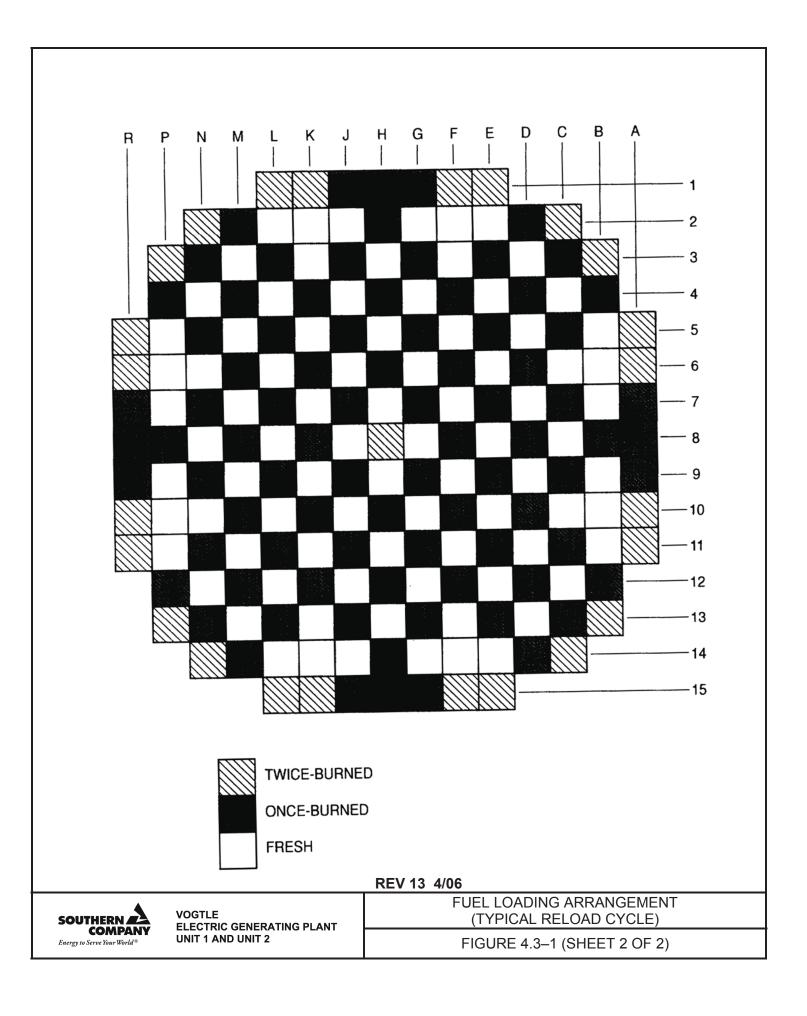
a. Isothermal coefficients, which include the Doppler effect in the fuel.

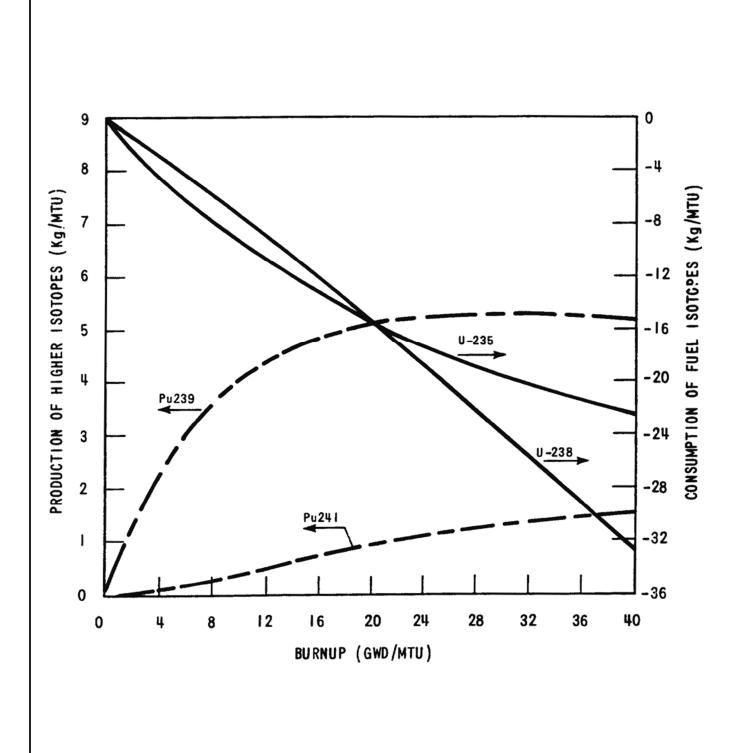
TABLE 4.3-12
BENCHMARK CRITICAL EXPERIMENTS

Description of Experiments ^(a)	Number of Experiments	LEOPARD k _{eff} Using Experimental Bucklings
UO ₂ Al clad SS clad Borated H ₂ O	14 19 7	1.0012 0.9963 0.9989
Subtotal	40	0.9985
U-Metal Al clad Unclad Subtotal	41 20 61	0.9995 0.9990 0.9993
Total	101	0.9990

a. Reported in reference 33.

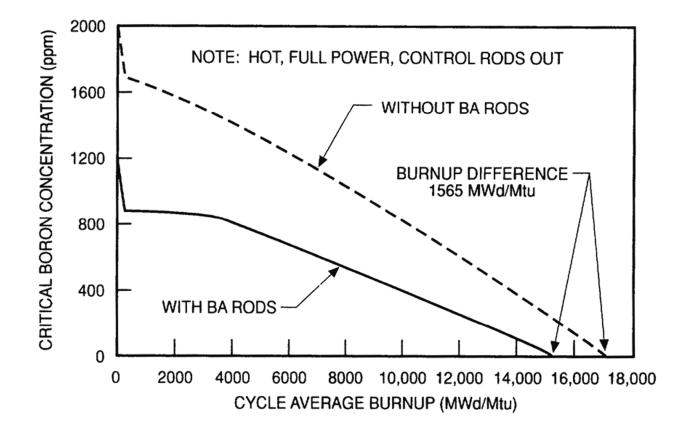








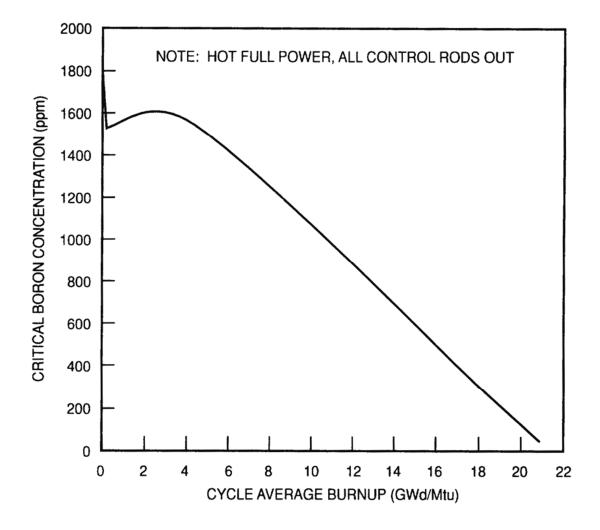
VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 PRODUCTION AND CONSUMPTION OF HIGHER ISOTOPES





VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 BORON CONCENTRATION VERSUS FIRST CYCLE BURNUP WITH AND WITHOUT BA RODS

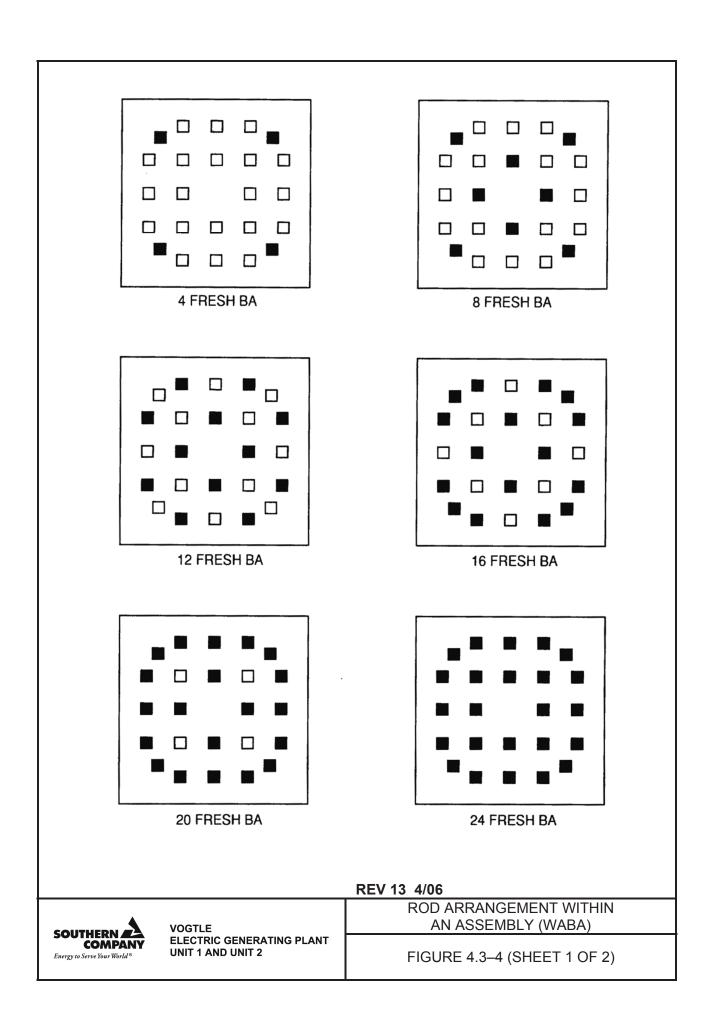
FIGURE 4.3-3 (SHEET 1 OF 2)

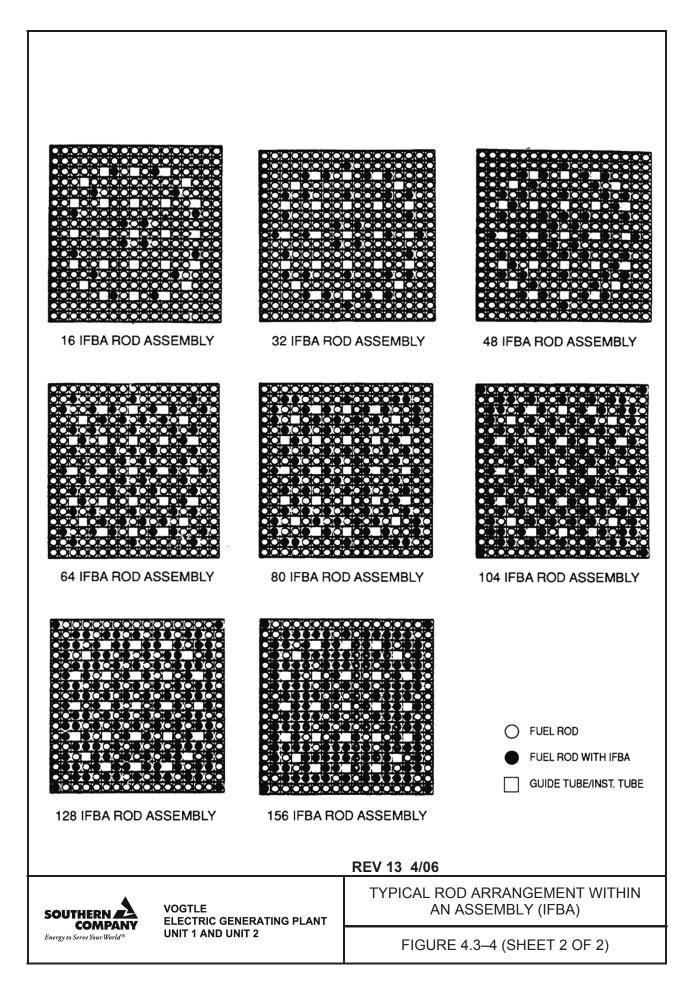




VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 BORON CONCENTRATION VERSUS RELOAD CYCLE BURNUP WITH INTEGRAL FUEL BURNABLE ABSORBERS

FIGURE 4.3-3 (SHEET 2 OF 2)





> NUMBER INDICATES NUMBER OF BURNABLE ABSORBER RODS S INDICATES SOURCE ROD

> > **REV 13 4/06**



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 LOADING PATTERN (TYPICAL DISCRETE BA)

FIGURE 4.3-5 (SHEET 1 OF 2)

F Ε D С Н G В Α R Κ 180° 4S 270° 4S 0°

NUMBER INDICATES NUMBER OF RODS WITH IFBA S INDICATES SOURCE ROD

REV 13 4/06



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 LOADING PATTERN (TYPICAL IFBA)

FIGURE 4.3-5 (SHEET 2 OF 2)

0.989	1.194	1.015	1.230	1.083	1.359	1.217	0.532
1.194	1.025	1.025 1.209 1.209 1.030 1.047 1.263		1.288	1.117	1.227	0.582
1.015	1.209	1.030	1.263	1.137	1.336	1.278	0.531
1.230	1.047	1.263	1.080	1.305	1.085	1.040	0.314
1.083	1.288	1.137	1.305	1.161	1.091	0.520	-
1.359	1.118	1.335	1.085	1.091	0.843	0.258	
1.217	1.222	1.273	1.039	0.520	0.258		
0.532	0.576	0.524	0.314				

ΑP

AVERAGE POWER

REV 13 4/06



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 NORMALIZED POWER DENSITY DISTRIBUTION NEAR BEGINNING OF LIFE, UNRODDED, HOT FULL POWER, NO XENON

1								
 1.024	1.218	1.035	1.242	1.087	1.345	1.201	0.535	_
1.218	1.047	1.227	1.058	1.288	1.111	1.209	0.582	
1.035	1.227	1.046	1.270	1.138	1.332	1.254	0.531	
1.242	1.059	1.289	1.086	1.299	1.078	1.027	0.317	
1.087	1.288	1.138	1.299	1.155	1.086	0.525		
1.345	1.111	1.321	1.079	1.086	0.844	0.265		
1.201	1.205	1.249	1.027	0.525	0.255			
0.535	0.577	0.525	0.318					

ΑP

ASSEMBLY POWER

REV 13 4/06



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 NORMALIZED POWER DENSITY DISTRIBUTION NEAR BEGINNING OF LIFE, UNRODDED, HOT FULL POWER, EQUILIBRIUM XENON

-	0.831	1.190	1.040	1.262	1.110	1.379	1.234	0.550	-
	1.190	1.037	1.236	1.072	1.310	1.133	1.239	0.598	
	1.040	1.236	1.054	1.277	1.141	1.333	1.273	0.541	
	1.262	1.072	1.277	1.071	1.264	1.060	1.028	0.320	
	1.110	1.310	1.141	1.253	1.045	1.037	0.512		
	1.379	1.134	1.332	1.060	1.037	0.807	0.255		
	1.234	1.235	1.269	1.028	0.513	0.255			
	0.550	0.593	0.535	0.320					

AP

ASSEMBLY POWER

REV 13 4/06



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 NORMALIZED POWER DENSITY DISTRIBUTION
NEAR BEGINNING OF LIFE, GROUP D AT ROD
INSERTION LIMITS, HOT FULL POWER,
EQUILIBRIUM XENON

ı								
 1.028	1.309	1.028	1.304	1.035	1.351	1.259	0.554	_
1.309	1.043	1.305	1.031	1.318	1.048	1.236	0.587	
1.028	1.305	1.027	1.316	1.075	1.329	1.190	0.517	
1.304	1.031	1.316	1.039	1.321	1.009	0.935	0.315	
1.035	1.319	1.076	1.321	1.082	1.128	0.532		ı
1.351	1.048	1.329	1.009	1.128	0.895	0.289		
1.259	1.234	1.188	0.986	0.533	0.289			
0.554	0.534	0.513	0.316					

ΑP

ASSEMBLY POWER

REV 13 4/06



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 NORMALIZED POWER DENSITY DISTRIBUTION NEAR MIDDLE OF LIFE, UNRODDED, HOT FULL POWER, EQUILIBRIUM XENON

1.252	1.001	1.247	1.004	1.275	1.223	0.606
1.013	1.249	1.003	1.256	1.019	1.213	0.642
1.249	1.001	1.257	1.044	1.280	1.186	0.583
1.003	1.257	1.015	1.280	1.023	1.045	0.376
1.256	1.044	1.280	1.095	1.190	0.821	
1.020	1.280	1.024	1.190	1.003	0.365	
1.212	1.185	1.046	0.621	0.365		
0.640	0.580	0.377				
	1.013 1.249 1.003 1.256 1.020	1.013 1.249 1.249 1.001 1.003 1.257 1.256 1.044 1.020 1.280 1.212 1.185	1.013 1.249 1.003 1.249 1.001 1.257 1.003 1.257 1.015 1.256 1.044 1.280 1.020 1.280 1.024 1.212 1.185 1.046	1.013 1.249 1.003 1.256 1.249 1.001 1.257 1.044 1.003 1.257 1.015 1.280 1.256 1.044 1.280 1.095 1.020 1.280 1.024 1.190 1.212 1.185 1.046 0.621	1.013 1.249 1.003 1.256 1.019 1.249 1.001 1.257 1.044 1.280 1.003 1.257 1.015 1.280 1.023 1.256 1.044 1.280 1.095 1.190 1.020 1.280 1.024 1.190 1.003 1.212 1.185 1.046 0.621 0.365	1.013 1.249 1.003 1.256 1.019 1.213 1.249 1.001 1.257 1.044 1.280 1.186 1.003 1.257 1.015 1.280 1.023 1.045 1.256 1.044 1.280 1.095 1.190 0.821 1.020 1.280 1.024 1.190 1.003 0.365 1.212 1.185 1.046 0.621 0.365

AP

ASSEMBLY POWER

REV 13 4/06



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 NORMALIZED POWER DENSITY DISTRIBUTION NEAR END OF LIFE, UNRODDED, HOT FULL POWER, EQUILIBRIUM XENON

0.912	1.227	1.009	1.271	1.025	1.307	1.257	0.625
1.227	1.008	1.262	1.018	1.280	1.040	1.243	0.660
1.009	1.262	1.011	1.257	1.047	1.290	1.203	0.595
1.271	1.019	1.257	1.002	1.241	1.004	1.045	0.379
1.025	1.280	1.047	1.242	0.054	1.136	0.506	
1.307	1.041	1.290	1.005	1.135	0.952	0.353	
1.257	1.241	1.202	1.046	0.507	0.353		
0.625	0.558	0.582	0.381				

AΡ

ASSEMBLY POWER

REV 13 4/06



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 NORMALIZED POWER DENSITY DISTRIBUTION NEAR END OF LIFE, GROUP D AT ROD INSERTION LIMITS, HOT FULL POWER, EQUILIBRIUM XENON

1.340	1.806	1.141	1.208	1.163	1.841	1.172	1.234	1.188	1.236	1.177	1.248	1.172	1.220	1.984	1.234	1.250
1.210	1.184	1 . 189	1.183	1.177	1.383	1.183	1.186	1.281	1.188	1.188	1.200	1.188	1.162	1.195	1.186	1.857
1.145	1.184	1 . 188	1.290	1.830		1.306	1.303		1.306	1.300		1.829	1.300	1.169	1.187	1.188
1.811	1.184	1.291		1.345	1.330	1.300	1.304	1.311	1.206	1.213	1.248	1.384		1.863	1.167	1.837
1.167	1 . 180	1.332	1.346	1.252	1.335	1.213	1.200	1.318	1.211	1.217	1.843	1.360	1.286	1.334	1.182	1.181
1.346	1.206		1.340	1.336		1.883	1.322		1.834	1.227		1.844	1.350		1.200	1.360
1.177	1.187	1.306	1.211	1.218	1.334	1.218	1.218	1.325	1.216	1.218	1.330	1.221	1.220	1.319	1.190	1.190
1.340	1.181	1.307	1.207	1.313	1.333	1.216	1.217	1.887	1.218	1.210	1.228	1.218	1.214	1.917	1.308	1.883
1.181	1.887		1.318	1.821		1.326	1.328		1.229	1.229		1.327	1.823		1.298	1.303
1.344	1.194	1.310	1.210	1.218	1.327	1.219	1.230	1.330	1.221	1.221	1.891	1.220	1.217	1.319	1.304	1.255
1.184	1.184	1.318	1.218	1.221	1.331	1.221	1.821	1.831	1.222	1.224	1.835	1.827	1.825	1.334	1.203	1.198
1.287	1.297		1.351	1.347		1.333	1.331		1.333	1.396		1.983	1.358		1.306	1.267
1.181	1 . 193	1 336	1.360	1.265	1.349	1.226	1.221	1.330	1.322	1.228	1.363	1.270	1.367	1.844	1.301	1.190
1.229	1.170	1.308		1.363	1.384	1.224	1.218	1.236	1.219	1.227	1.300	1.908		1.918	1.178	1.230
1 163	1.143	1.177	1.310	1.841		1.324	1.821		1.883	1.326		1.348	1.216	1 . 183	1.180	1.171
1.894	1.166	1,148	1.174	1.196	1.300	1.204	1.306	1.302	1.307	1.304	1.300	1.302	1.178	1.190	1.162	1.343
1.367	1.235	1.166	1.233	1.187	1.368	1.196	1.296	1.206	1.887	1.196	1.300	1.191	1.236	1.171	1.841	1.878

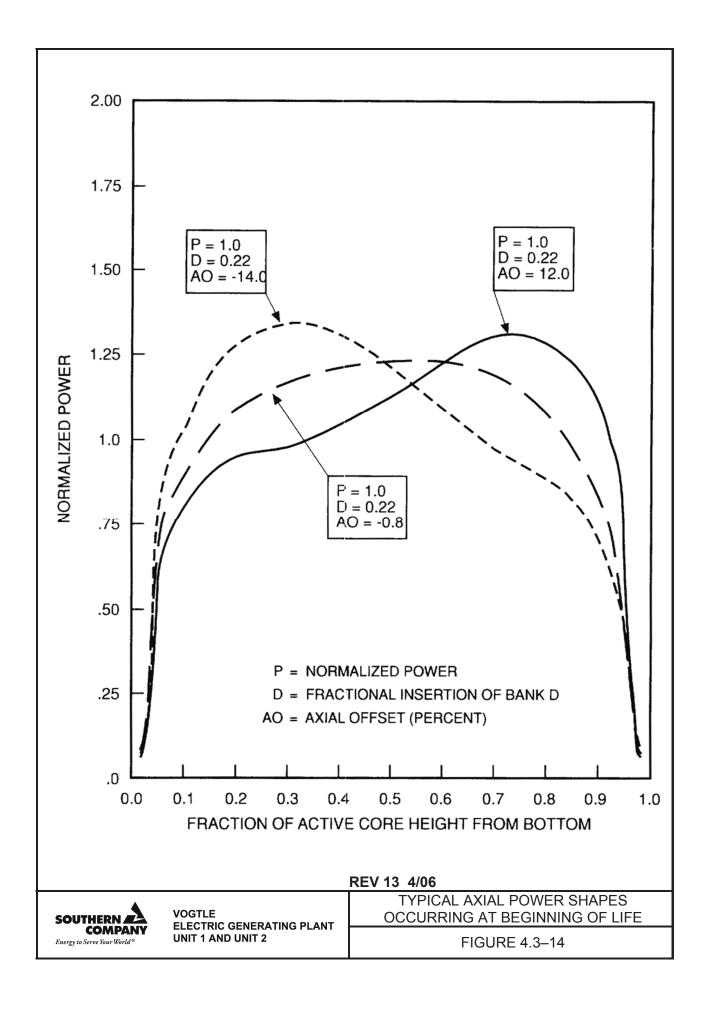


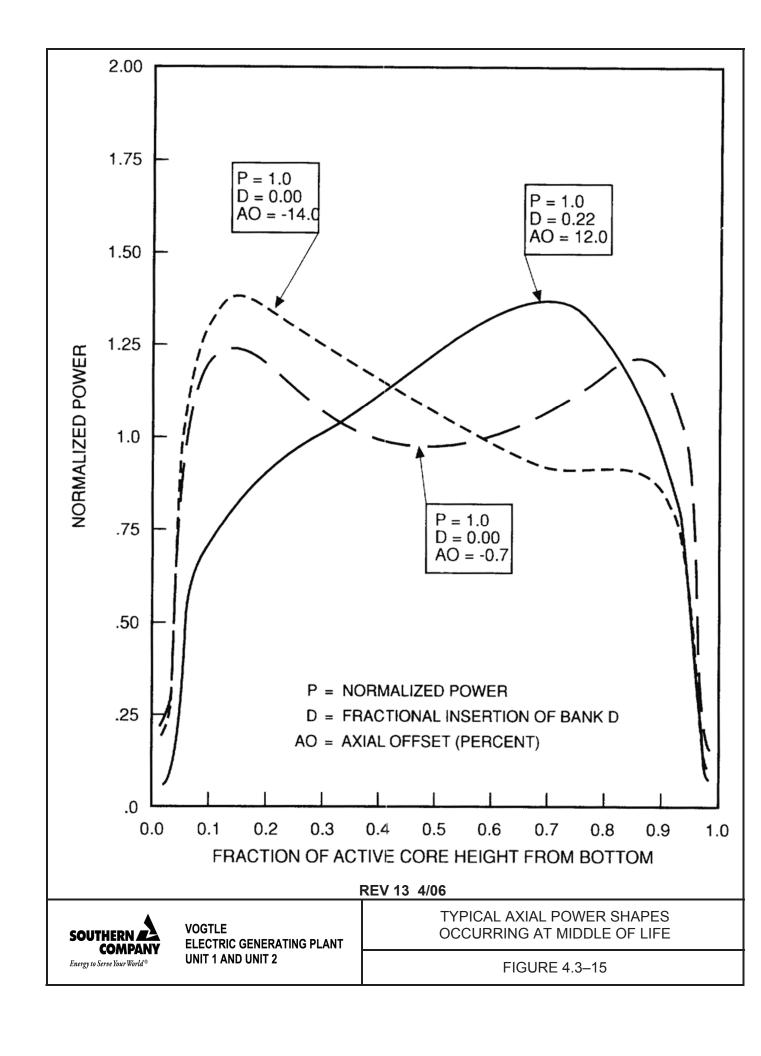
VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 RODWISE POWER DISTRIBUTION IN A TYPICAL ASSEMBLY (G-10) NEAR BEGINNING OF LIFE, HOT FULL POWER, EQUILIBRIUM XENON, UNRODDED CORE

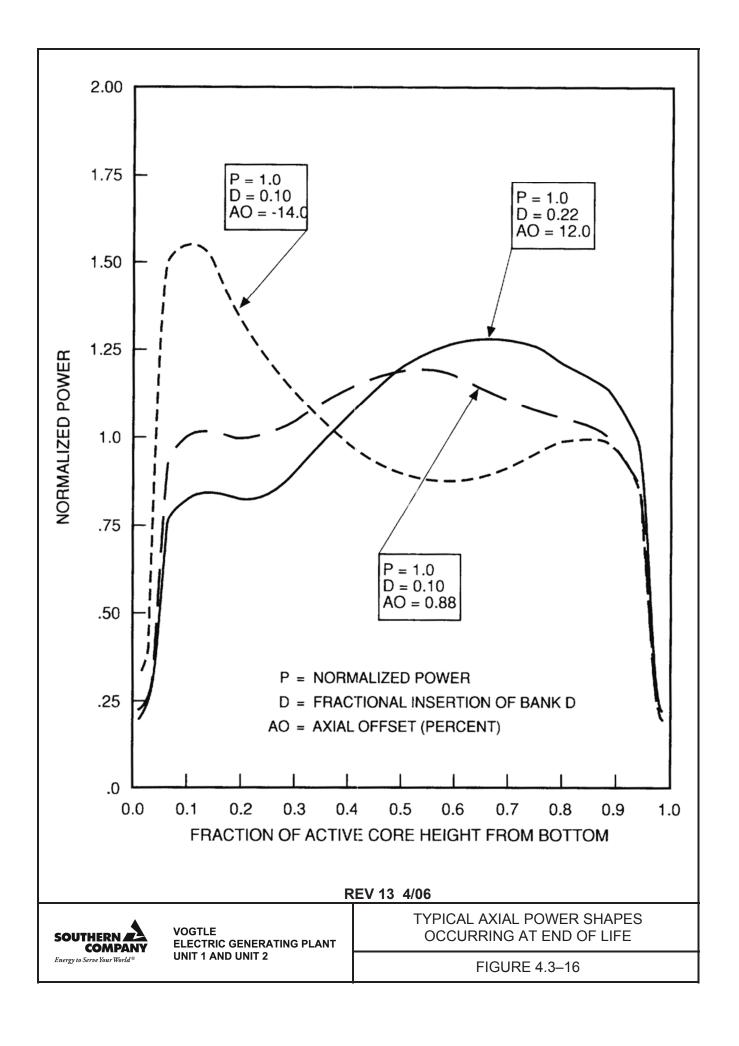
					_											
1.218	1.180	1.218	1.208	1.237	1.226	1.244	1.223	1.280	1.223	1.244	1.827	1.238	1,306	1.219	1.200	1.230
1.100	1.188	1.302	1.220	1.237	1.346	1.243	1.243	1.247	1.243	1.343	1.246	1.336	1.221	1.203	1.800	1.200
1.218	1.302	1.222	1.347	1.364		1.238	1.287		1.257	1.258		1.365	1.848	1.223	1.204	1.230
1.206	1.220	1.847		1.277	1.278	1.258	1.256	1.362	1.255	1.250	1.276	1.278		1.248	1.221	1.205
1.237	1.237	1.264	1.277	1.281	1.278	1.261	1.280	1.267	1.259	1.263	1.276	1.282	1.278	1.368	1.220	1.228
1.226	1.346		1.278	1.278		1.370	1.370		1.270	1.271		1.276	1.276		1.247	1.238
1.244	1.343	1.258	1.258	1.261	1.270	1.363	1.263	1.272	1.864	1.863	1.271	1.362	1.250	1.250	1.344	1.345
1.223	1.243	1.287	1.255	1.259	1.270	1.363	1.264	1.373	1.268	1.364	1.370	1.260	1.296	1.258	1.248	1.225
1.251	1.247		1.262	1.267		1.272	1.273		1.273	1.272		1.268	1.263		1.248	1.252
1.223	1.343	1.287	1.253	1.269	1.270	1.264	1.365	1.273	1.265	1.264	1.271	1.360	1.296	1'.258	1.245	1.225
1.344	1.244	1.258	1.288	1.262	1.271	1.364	1.264	1.272	1.264	1.364	1.272	1.263	1.250	1.260	1.248	1.346
1.227	1.247		1.276	1.276		1.271	1.271		1.871	1.272		1.877	1.277		1.848	1.229
1.238	1.238	1.265	1.278	1.882	1.276	1.262	1.260	1.368	1.260	1.263	1.277	1.283	1.279	1.266	1.340	1.240
1.306	1.321	1.248		1.278	1.276	1.250	1.256	1.263	1.296	1.260	1.877	1.279		1.349	1.223	1.200
1.230	1.204	1.223	1.248	1.265		1.250	1.258		1.258	1.200		1.306	1.249	1.225	1.206	1.222
1.300	1.200	1.204	1.221	1.339	1.347	1.244	1.245	1.348	1.348	1.245	1.248	1.340	1.822	1.305	1.302	1.202
1.219	1.200	1.218	1.206	1.888	1.227	1.346	1.224	1.251	1.224	1.848	1.830	1.220	1.307	1.221	1.802	1.221



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 RODWISE POWER DISTRIBUTION IN A TYPICAL ASSEMBLY (G-10) NEAR THE END OF LIFE, HOT FULL POWER, EQUILIBRIUM XENON, UNRODDED CORE







DELETED

REV 13 4/06



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 COMPARISON OF A TYPICAL ASSEMBLY AXIAL POWER DISTRIBUTION WITH CORE AVERAGE AXIAL DISTRIBUTION, BANK D SLIGHTLY INSERTED

DELETED

REV 13 4/06



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 FLOW CHART FOR DETERMINING SPIKE MODEL

DELETED

REV 13 4/06



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 PREDICTED POWER SPIKE DUE TO SINGLE NONFLATTENED GAP IN THE ADJACENT FUEL

DELETED

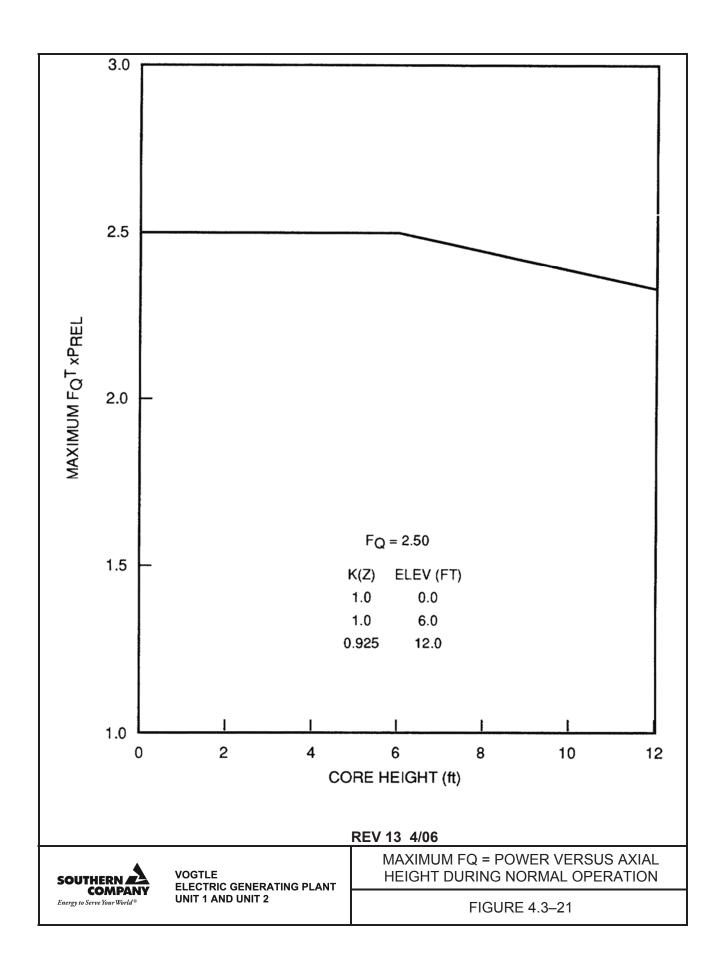
REV 13 4/06

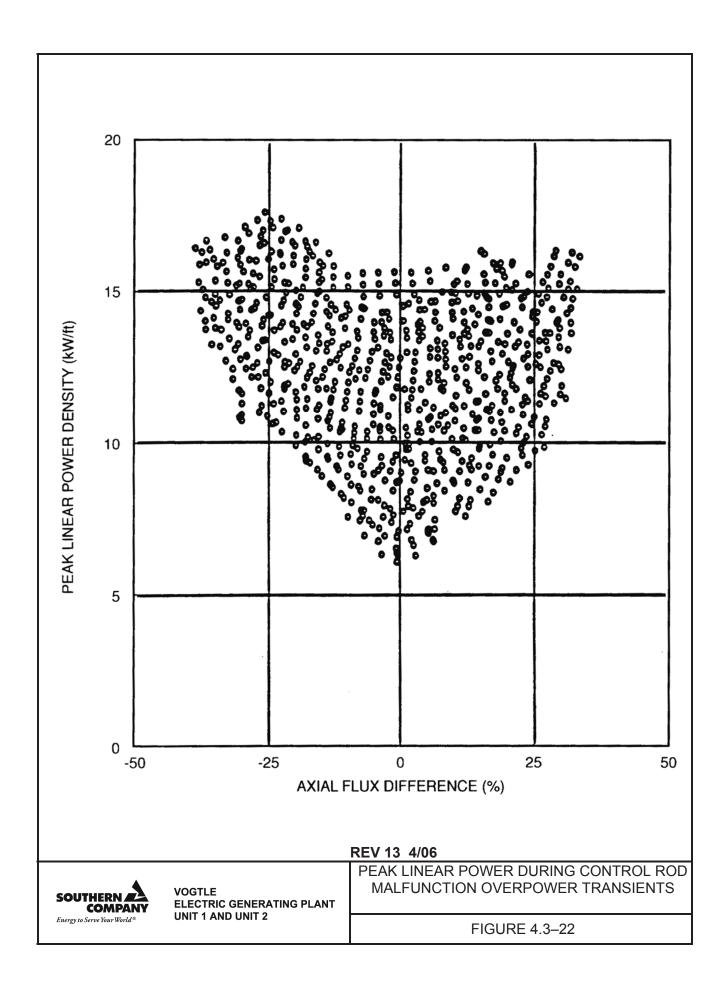
POWER SPIKE FACTOR AS A FUNCTION OF AXIAL POSITION

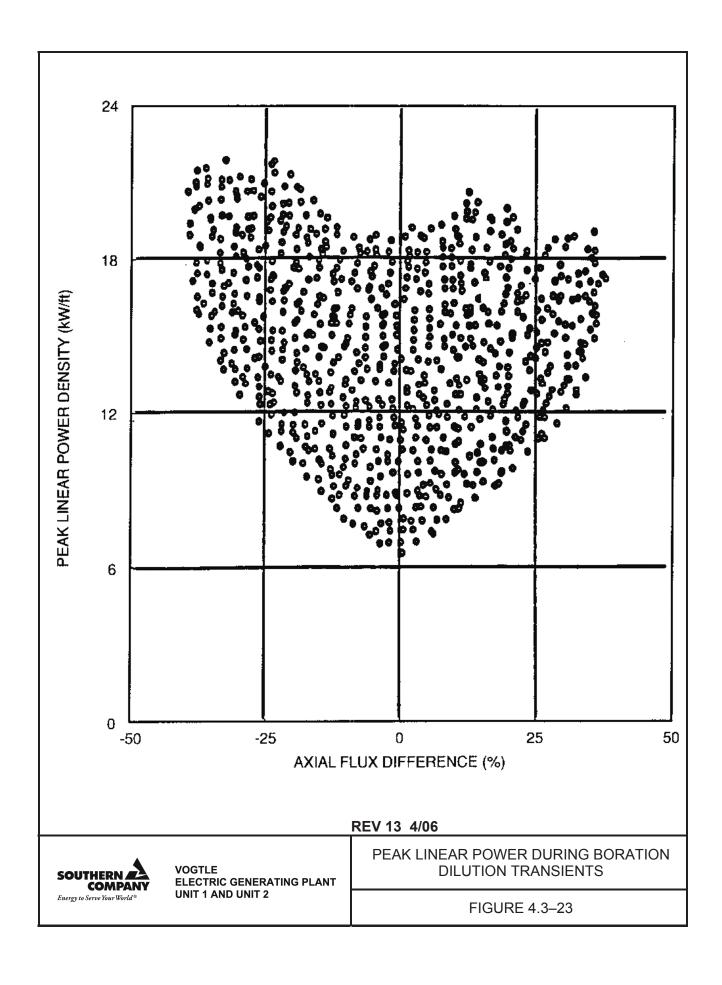
FIGURE 4.3-20

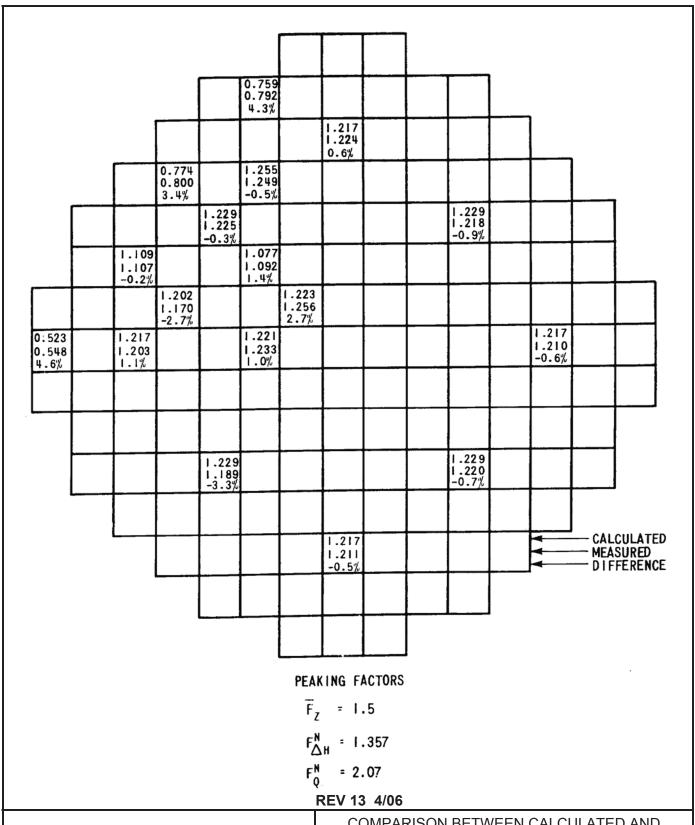


VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2



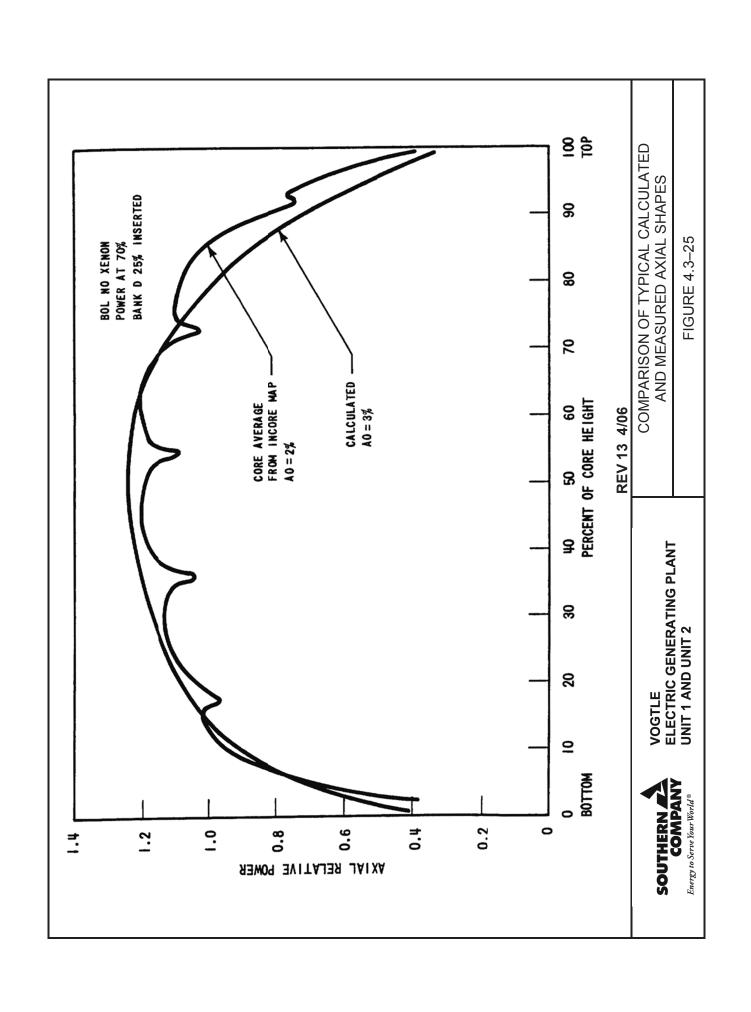


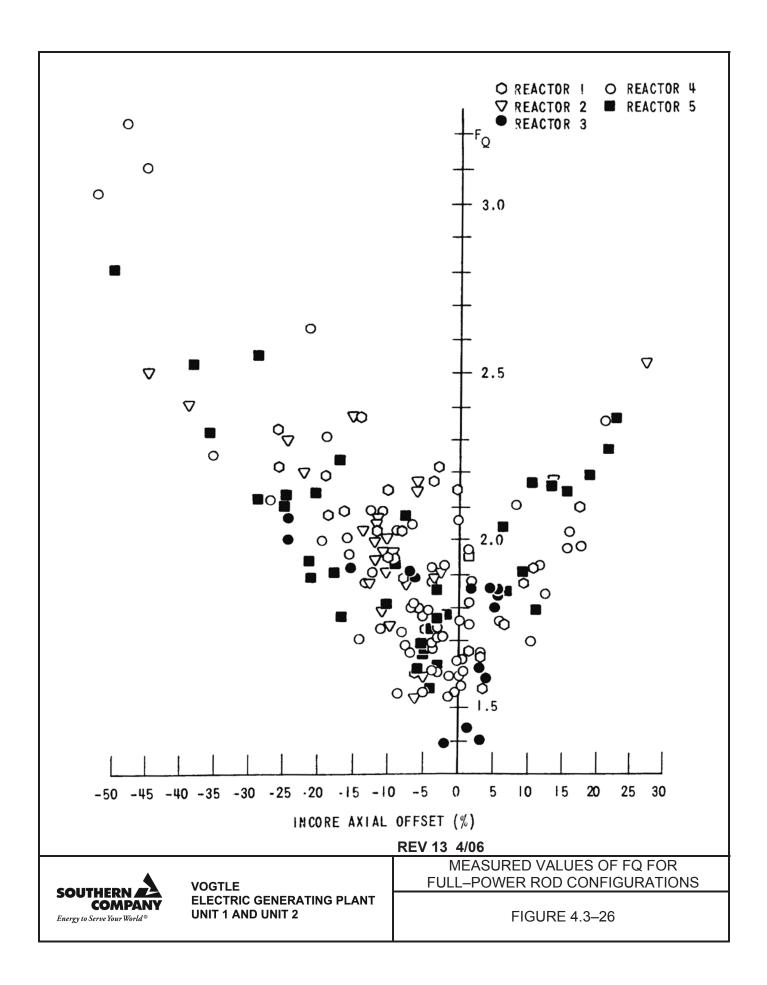


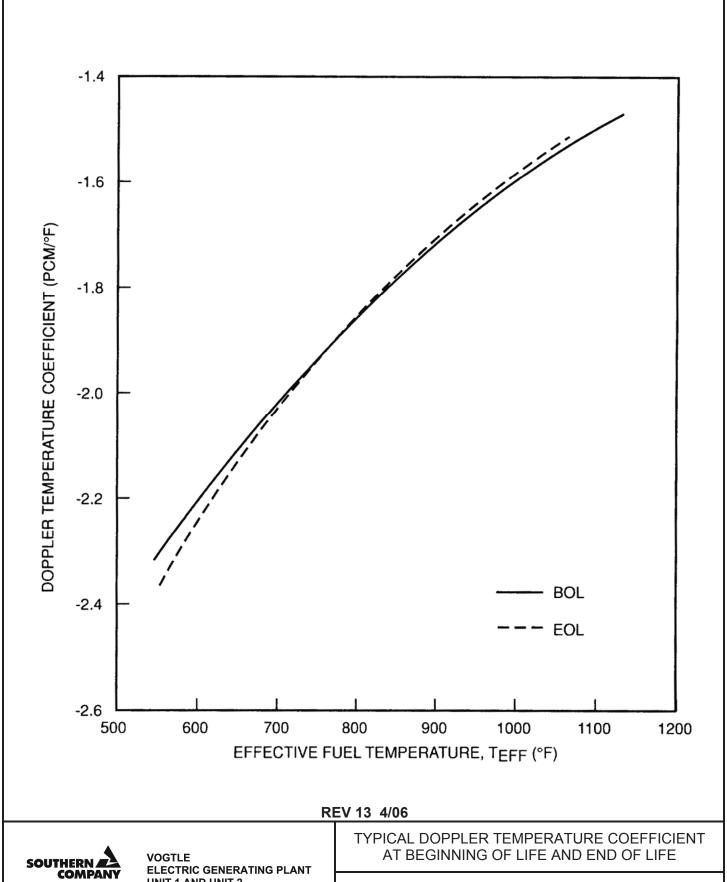




VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 COMPARISON BETWEEN CALCULATED AND MEASURED RELATIVE FUEL ASSEMBLY POWER DISTRIBUTION

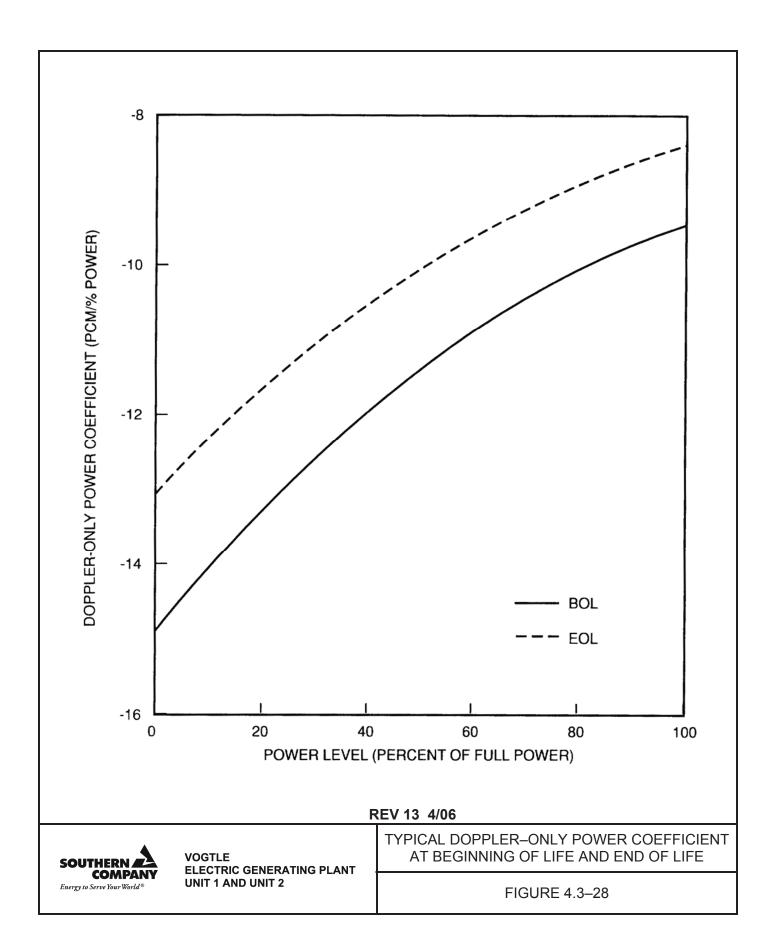


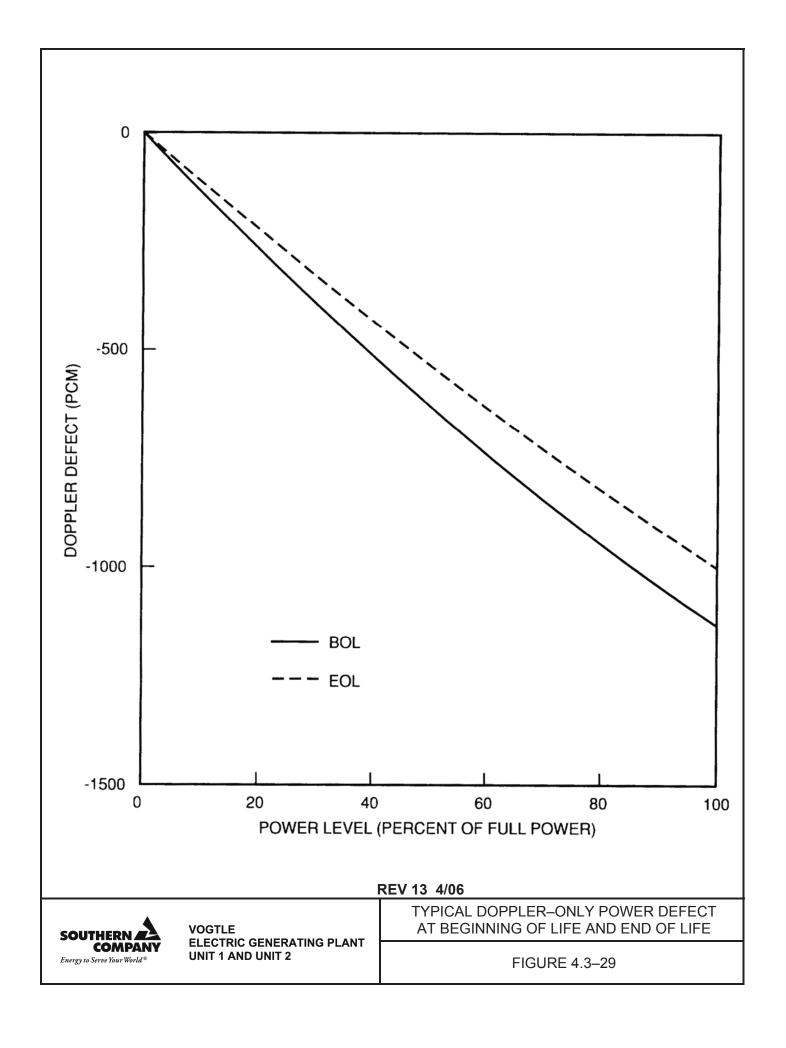


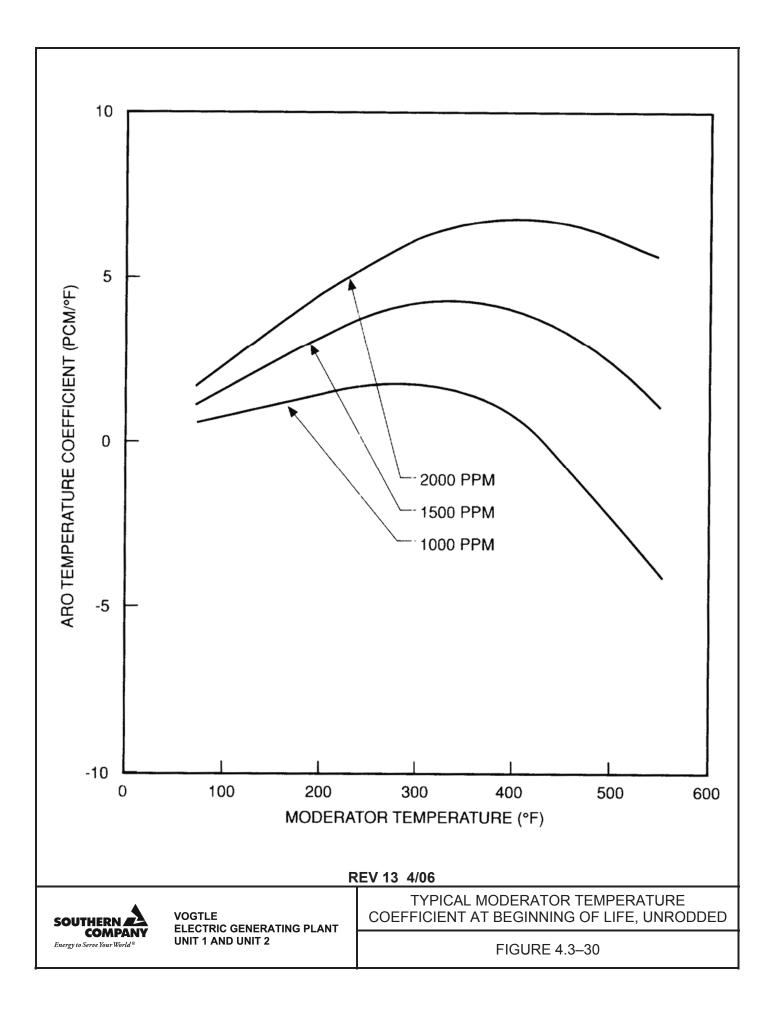


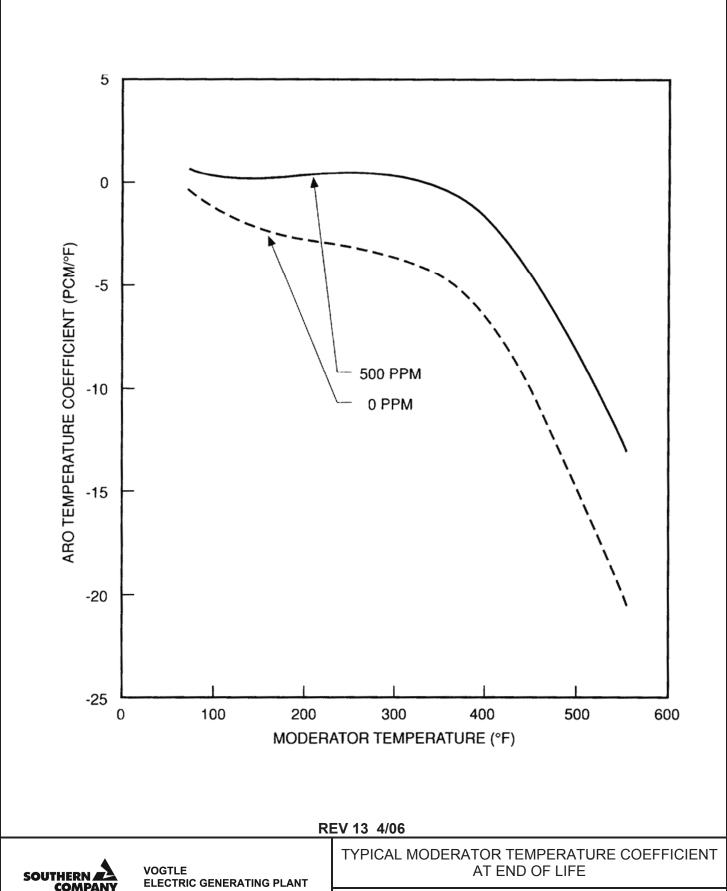
Energy to Serve Your World®

UNIT 1 AND UNIT 2



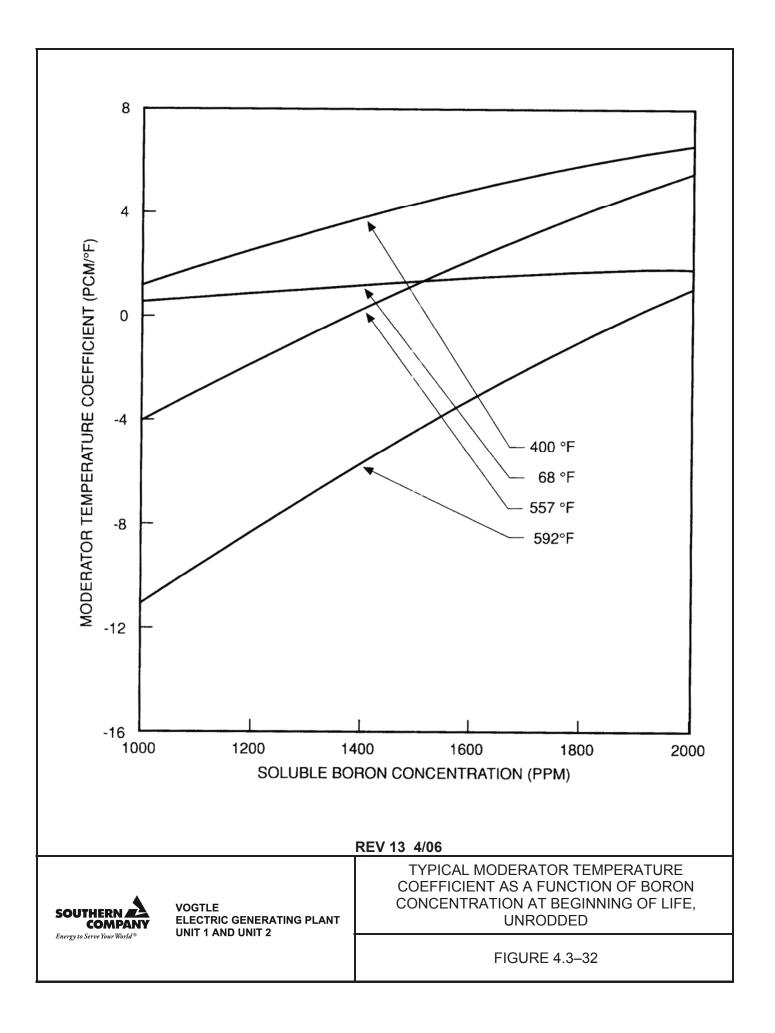


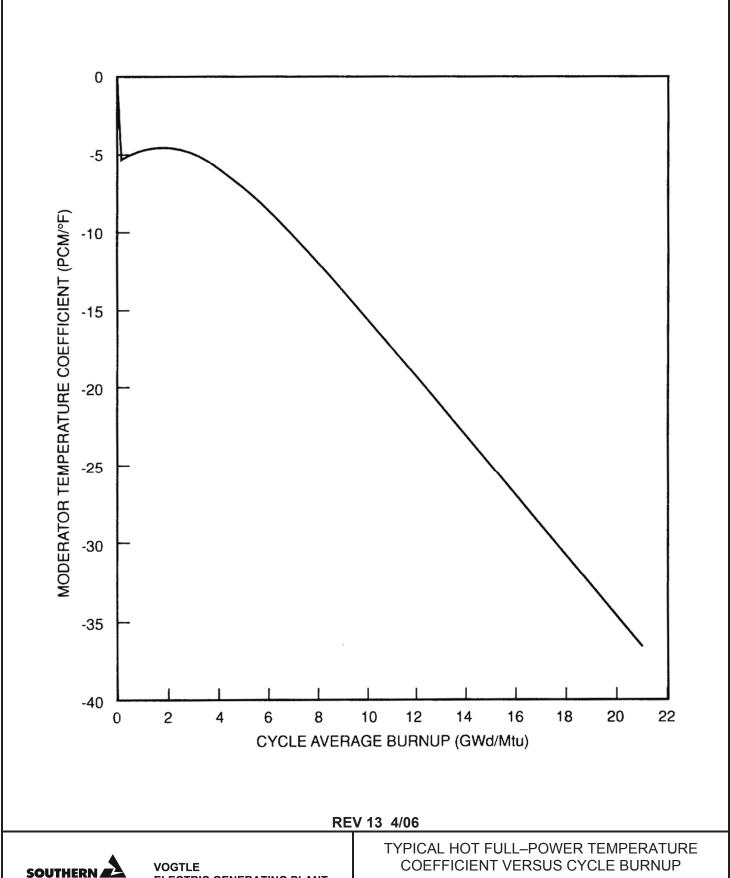




Energy to Serve Your World®

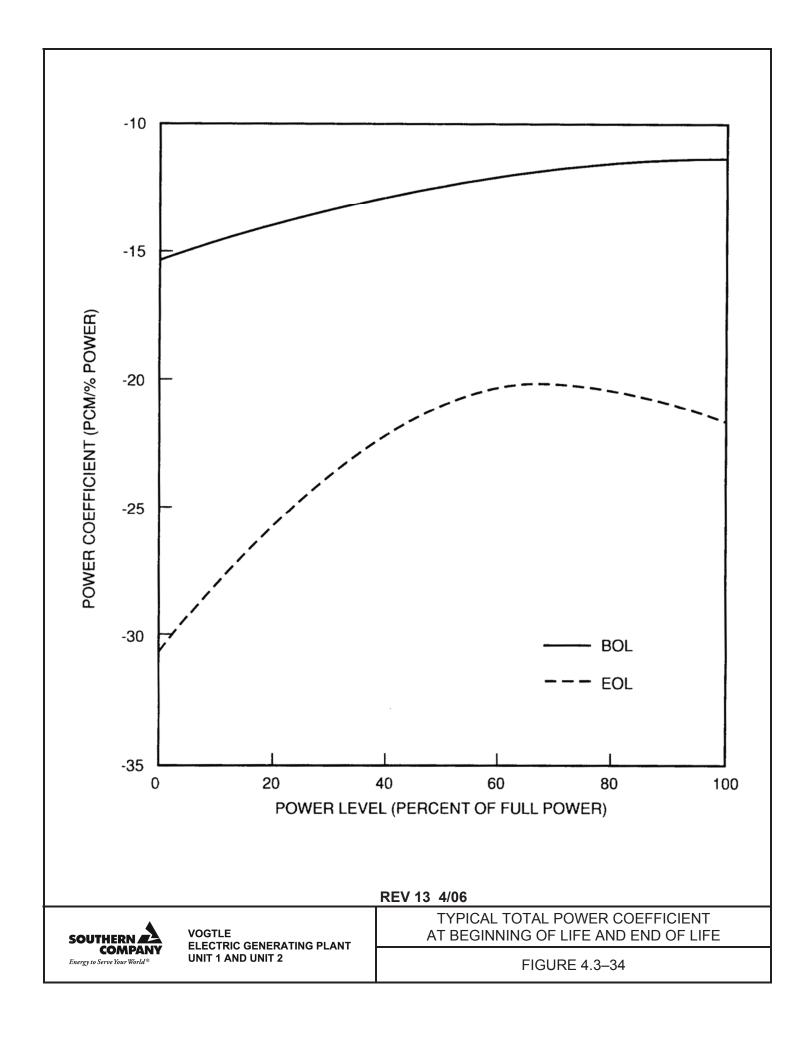
UNIT 1 AND UNIT 2

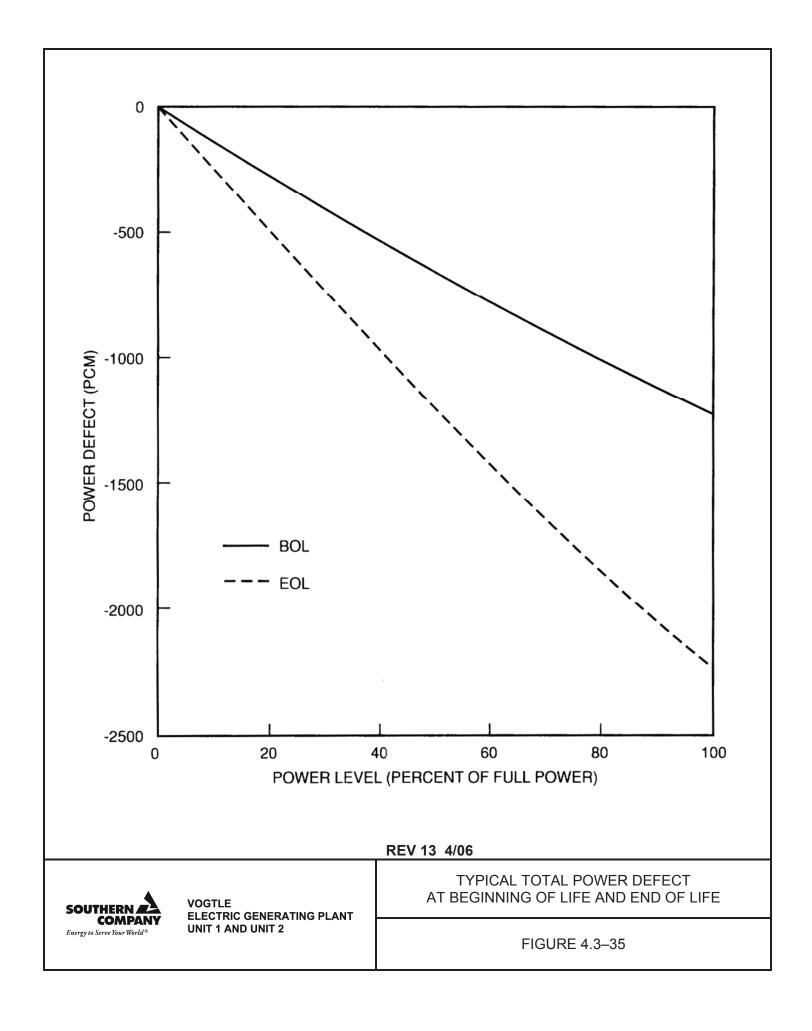


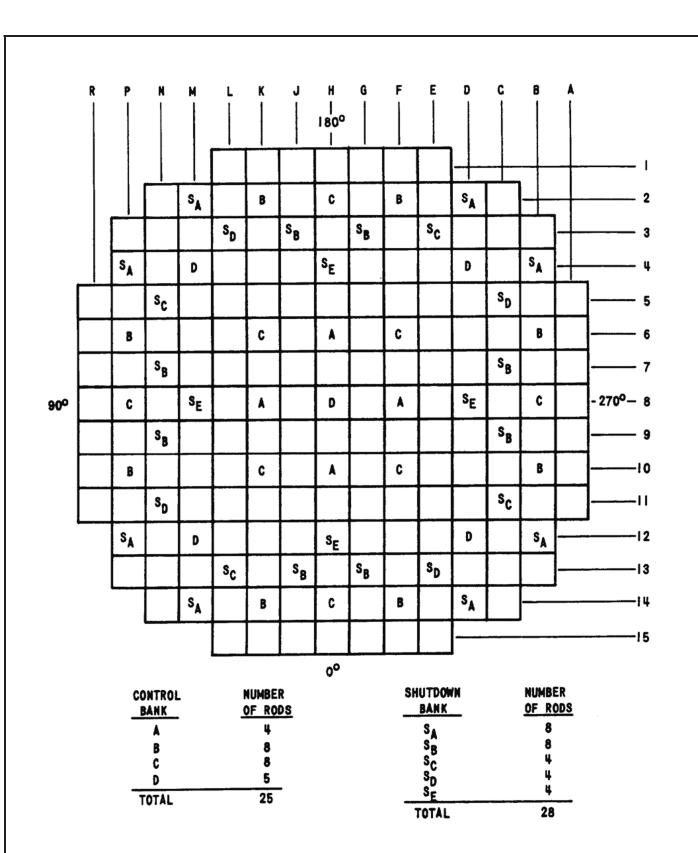


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ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2

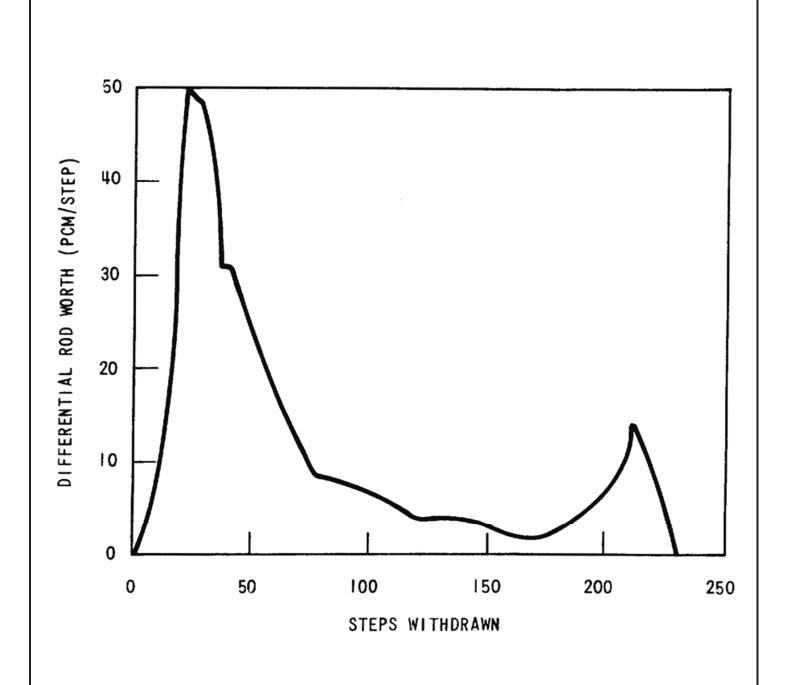






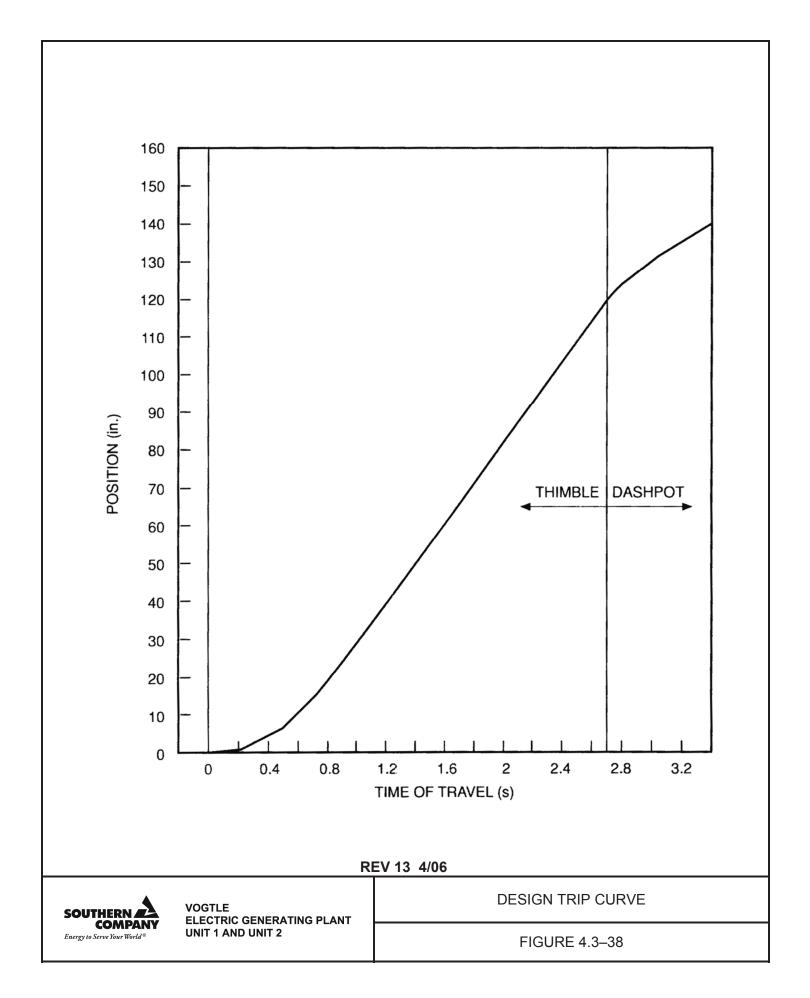


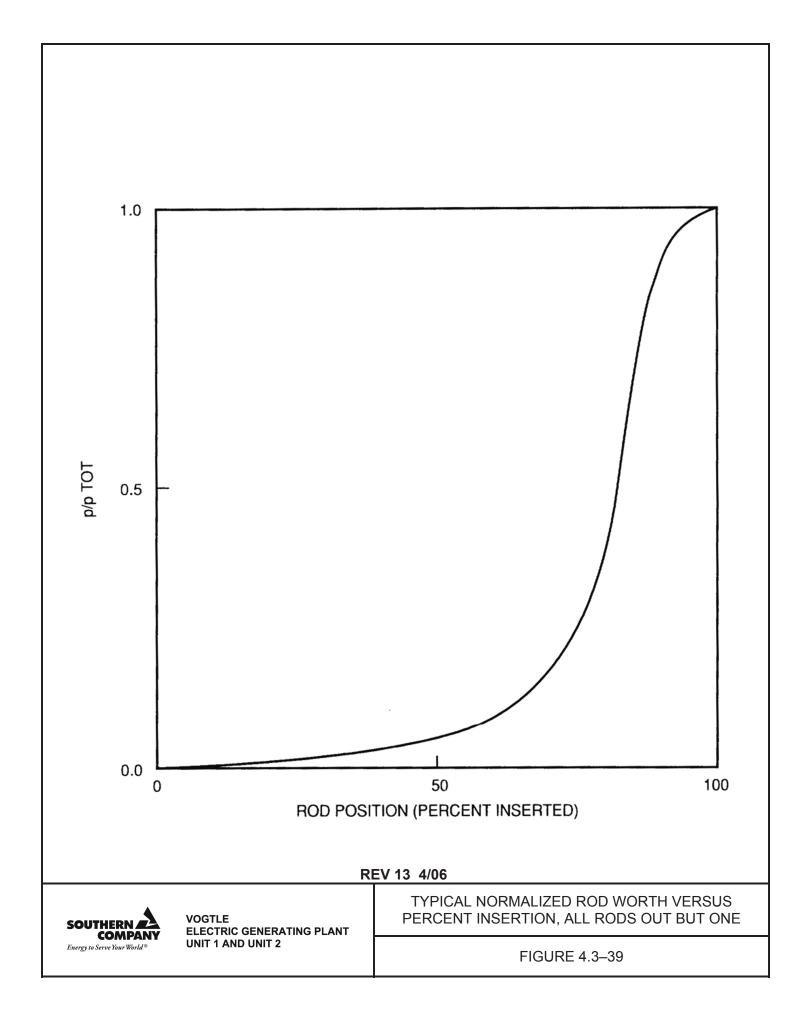
VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 ROD CLUSTER CONTROL ASSEMBLY PATTERN

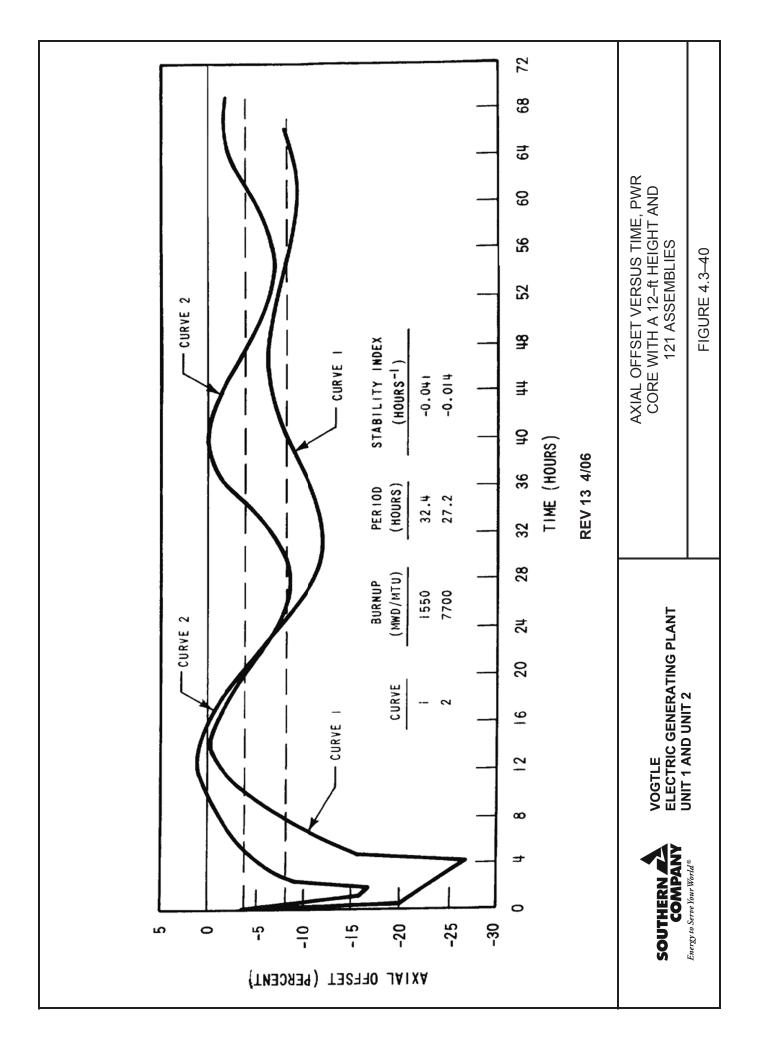


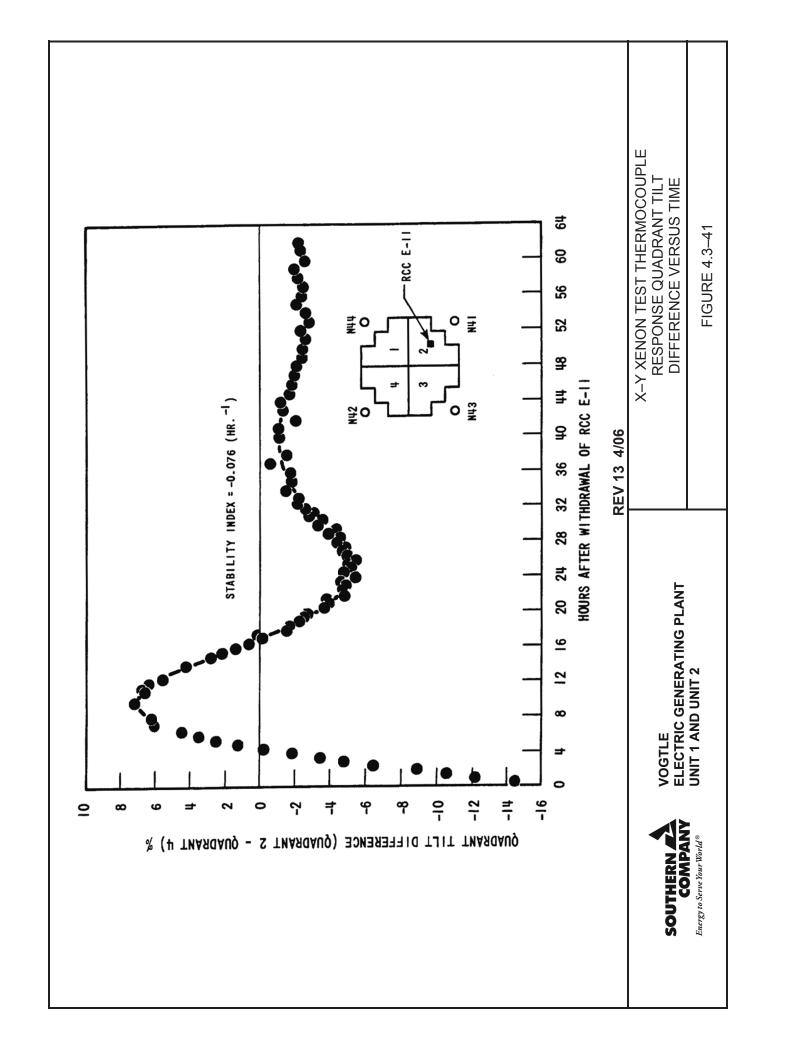


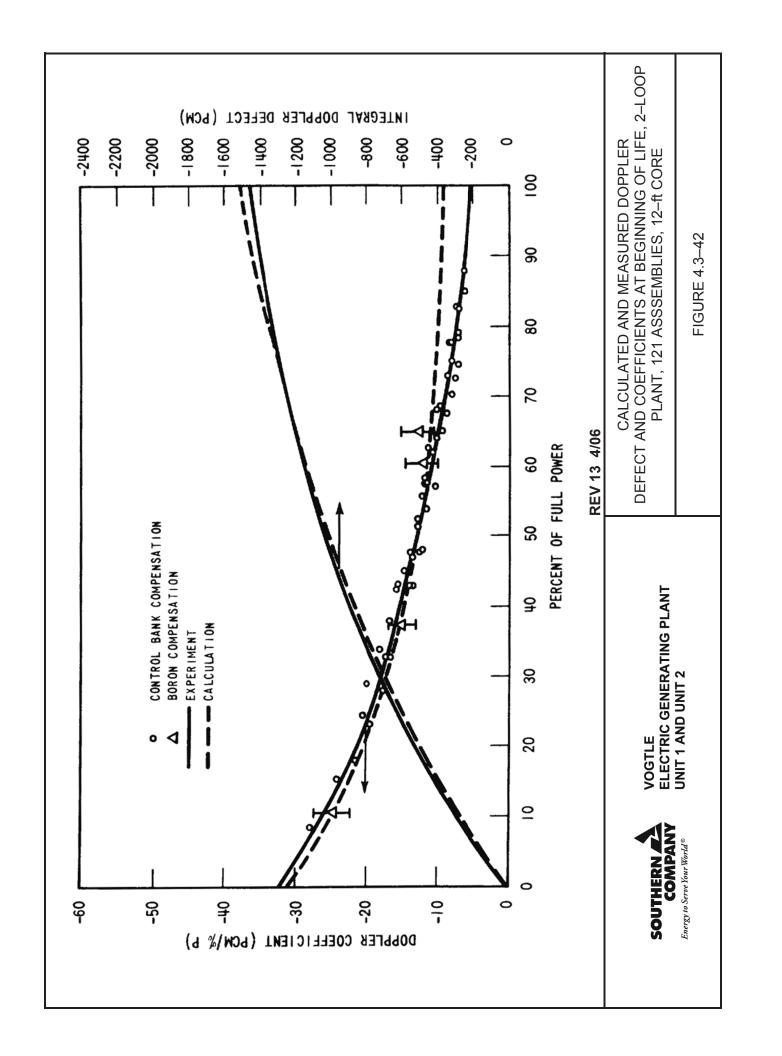
VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 TYPICAL ACCIDENTAL SIMULTANEOUS WITHDRAWAL OF TWO CONTROL BANKS AT END OF LIFE, HOT ZERO POWER, BANK D AND B MOVING IN THE SAME PLANE

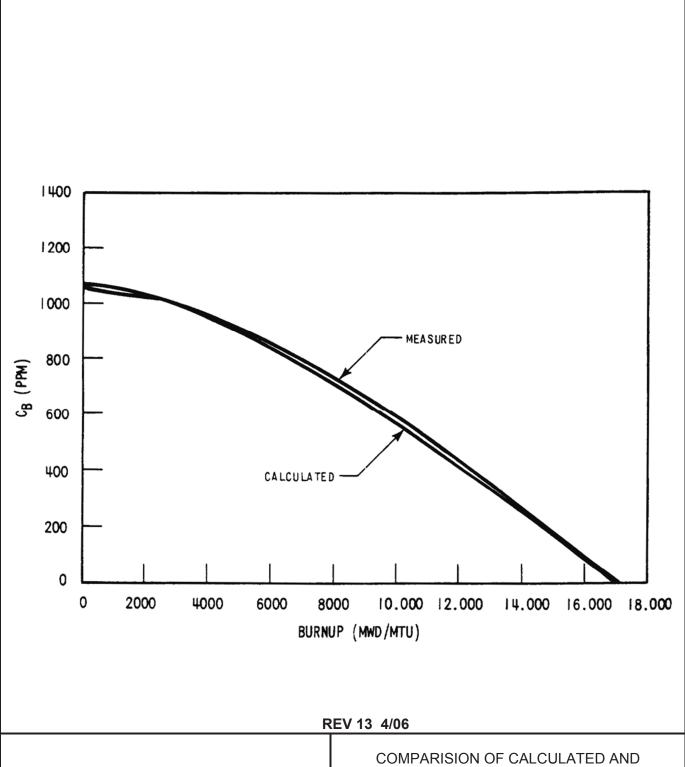






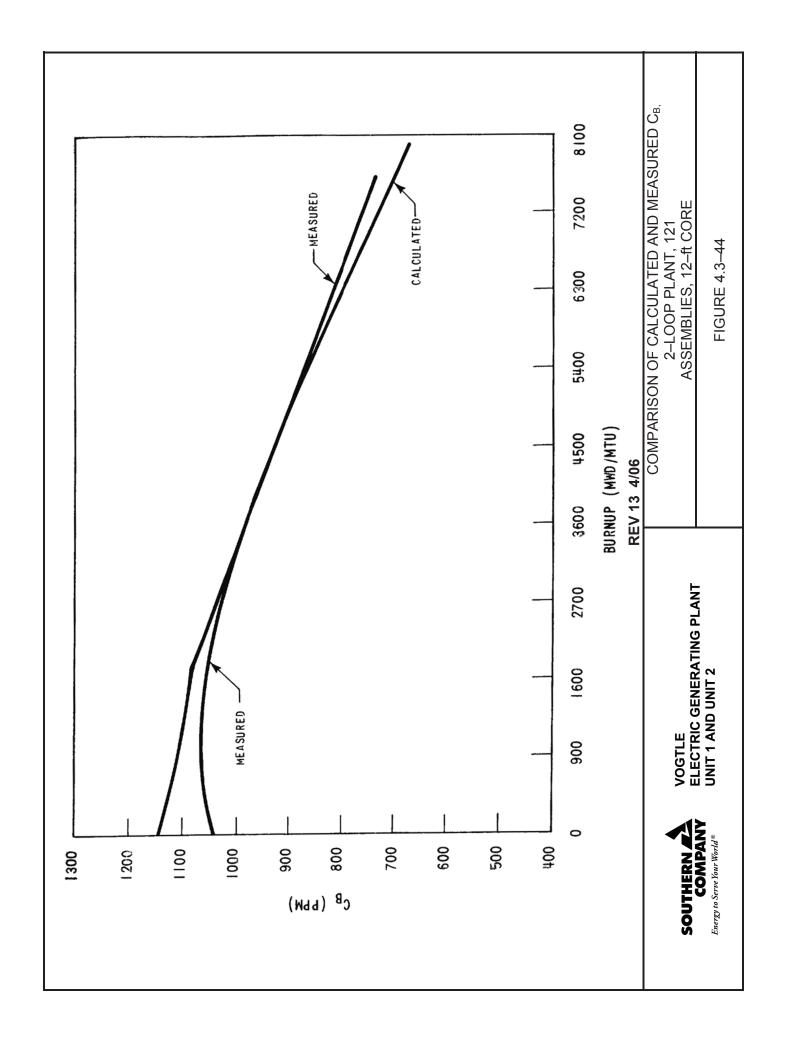


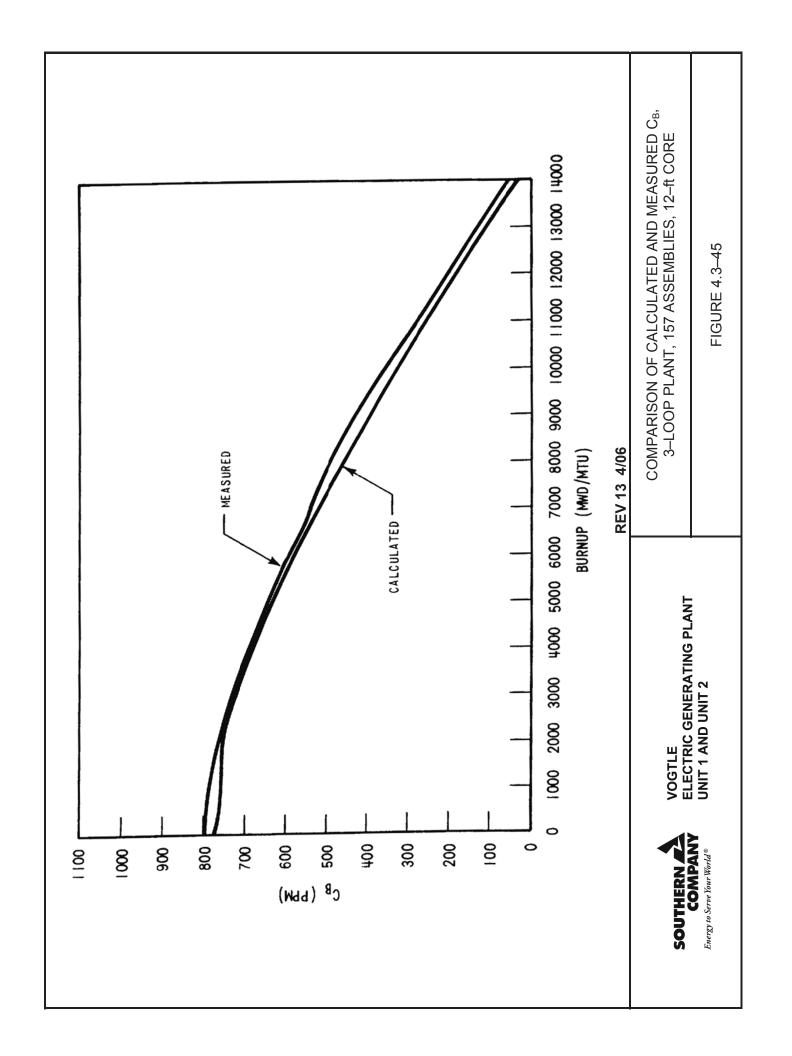


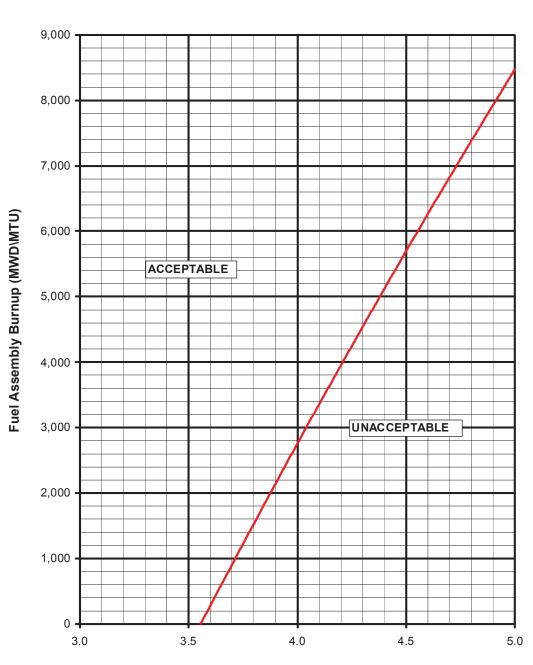




VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 COMPARISION OF CALCULATED AND MEASURED BORON CONCENTRATION, 2-LOOP PLANT, 121 ASSEMBLIES, 12-ft CORE



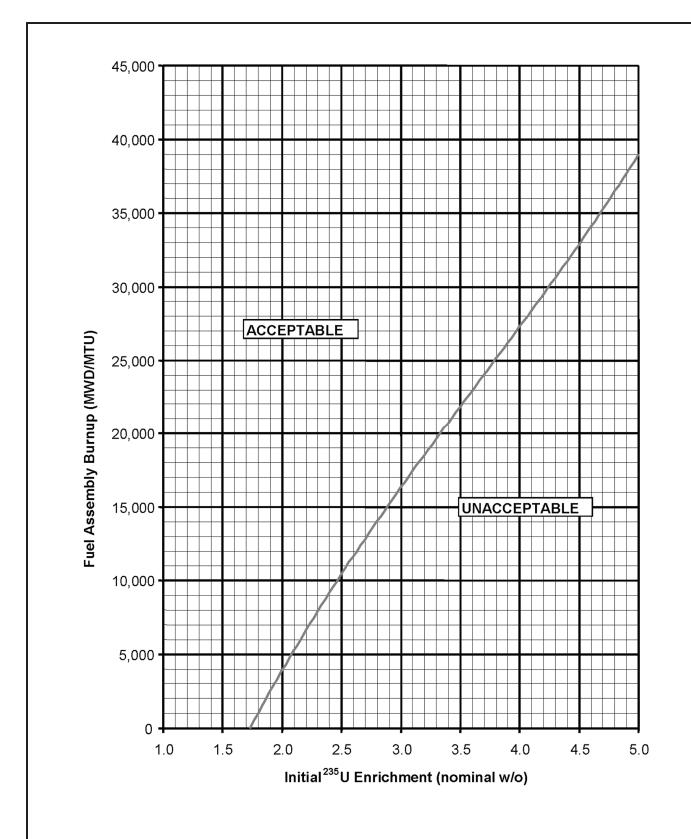




Initial ²³⁵U Enrichment (nominal w/o)

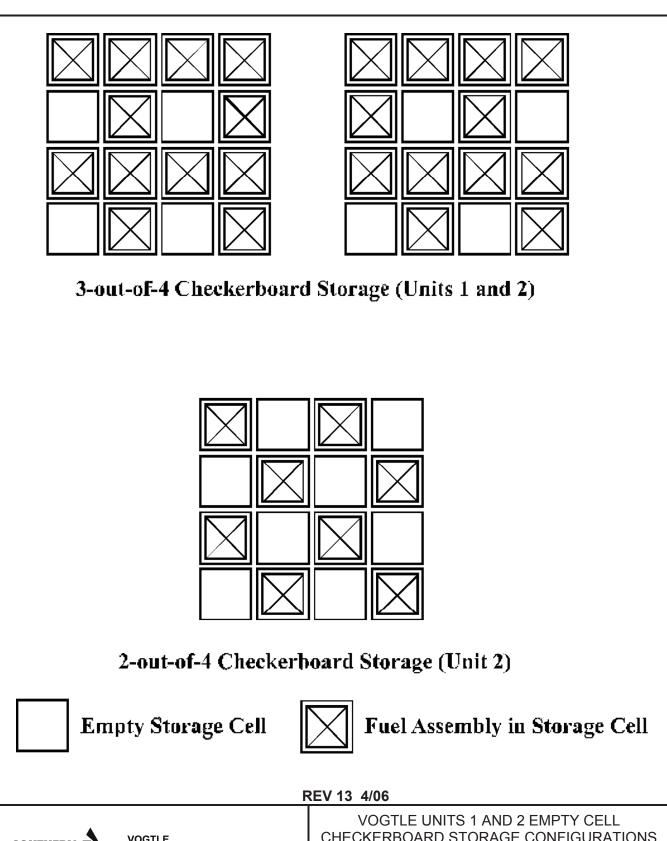


VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 VOGTLE UNIT 1 BURNUP CREDIT REQUIREMENTS FOR ALL CELL STORAGE





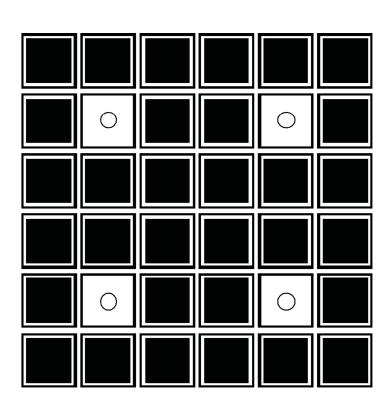
VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 VOGTLE UNIT 2 BURNUP CREDIT REQUIREMENTS FOR ALL CELL STORAGE



SOUTHERN **COMPANY**

ELECTRIC GENERATING PLANT **UNIT 1 AND UNIT 2**

CHECKERBOARD STORAGE CONFIGURATIONS



3x3 Checkerboard Storage



Low Enrichment Fuel Assembly in Storage Cell

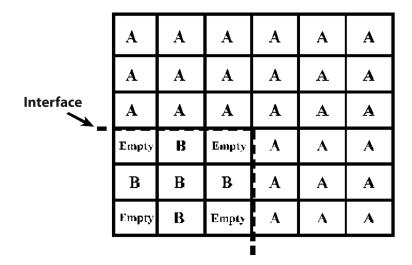


High Enrichment Fuel Assembly in Storage Cell

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VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 VOGTLE UNIT 2 3X3 CHECKERBOARD STORAGE CONFIGURATION



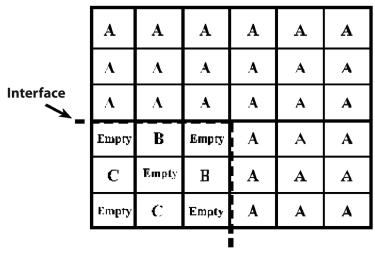
Note:

A = All Cell Enrichment

B = 3-Out-of-4 Enrichment

Empty = Empty Cell

Boundary Between All Cell Storage and 3-Out-of-4 Storage (Units 1 and 2)



Note:

A = All Cell Enrichment

B = 3-Out-of-4

Enrichment

C= 2-Out-of-4

Enrichment Empty = Empty Cell

Boundary Between All Cell Storage and 2-Out-of-Storage (Unit 2)

Note:

- 1. A row of empty cells can be used at the interface to separate the configurations.
- 2. It is acceptable to replace an assembly with an empty cell.

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VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 VOGTLE UNITS 1 AND 2 INTERFACE REQUIREMENTS (ALL CELL TO CHECKERBOARD STORAGE)

	В	Empty	В	Empty	В	Empty
	В	В	В	В	В	В
Interface	В	Empty	В	Empty	В	Empty
_	Emply	C	Empty	В	В	В
	C	Empty	С	Empty	В	Empty
	Empty	C	Empty	В	В	В

Note: B = 3-Out-of-4 Enrichment C = 2-Out-of-4 Enrichment

Empty = Empty Cell

Boundary Between 2-Out-of-4 Storage and 3-Out-of-4 Storage

В B B Empty В Empty B В Empty В В В Interface Empty В B В В Empty \mathbf{C} C **Empty** В Empty Empty Empty Empty \mathbf{C} R R В В **Empty** Empty **Empty**

Note:
B = 3-Out-of-4
Enrichment
C = 2-Out-of-4
Enrichment
Empty = Empty Cell

Boundary Between 2-Out-of-4 Storage and 3-Out-of-4 Storage

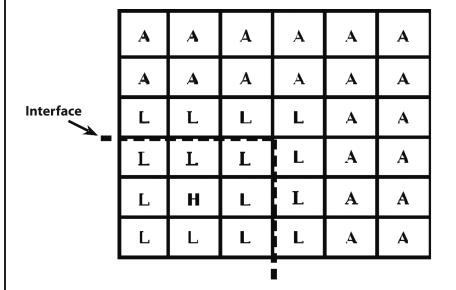
Note:

- 1. A row of empty cells can be used at the interface to separate the configurations.
- 2. It is acceptable to replace an assembly with an empty cell.

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VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 VOGTLE UNIT 2 INTERFACE REQUIREMENTS (CHECKERBOARD STORAGE INTERFACE)



Note:

A = All Cell Enrichment
L = Low Enrichment of
3 x 3 Checkerboard
H = High Enrichment of
3 x 3 Checkerboard

Note:

- 1. A row of empty cells can be used at the interface to separate the configurations.
- 2. It is acceptable to replace an assembly with an empty cell.

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VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 VOGTLE UNIT 2 INTERFACE REQUIREMENTS (3X3 CHECKERBOARD TO ALL CELL STORAGE)

	В	В	В	В	В	в*
	Empty	В	Empty	В	F.mpty	В
Interface	L	I.	L	L	В	В
_	L	L	Ī,	L	Empty	В
	L	Н	L	ī.	В	В
	L	L	L	L	lsmpty	В

Note:

B = 3-Out-of-4 Enrichment

L= Low Enrichment of 3x3 Storage

H = High Enrichment of 3x3 Storage Empty = Empty Cell

Boundary Between 3x3 Storage and 3-Out-of-4 Storage

	С	Empty	C	Kmpty	С	Empty
	Empty	В	Empty	В	Empty	С
Interface	L	Ľ**	L	l.**	В	Empty
_	L	L	L	L	Empty	C
	L	н	L	l.**	В	Empty
	L	L	L	L	Empty	C
•						

Note:

B = 3-Out-of-4 Enrichment

L = Low Enrichment of 3x3 Storage

H= High Enrichment of 3x3 Storage

C= 2-Out-of-4 Enrichment Empty = Empty Cell

Boundary Between 3x3 Storage and 2-Out-of-4 Storage

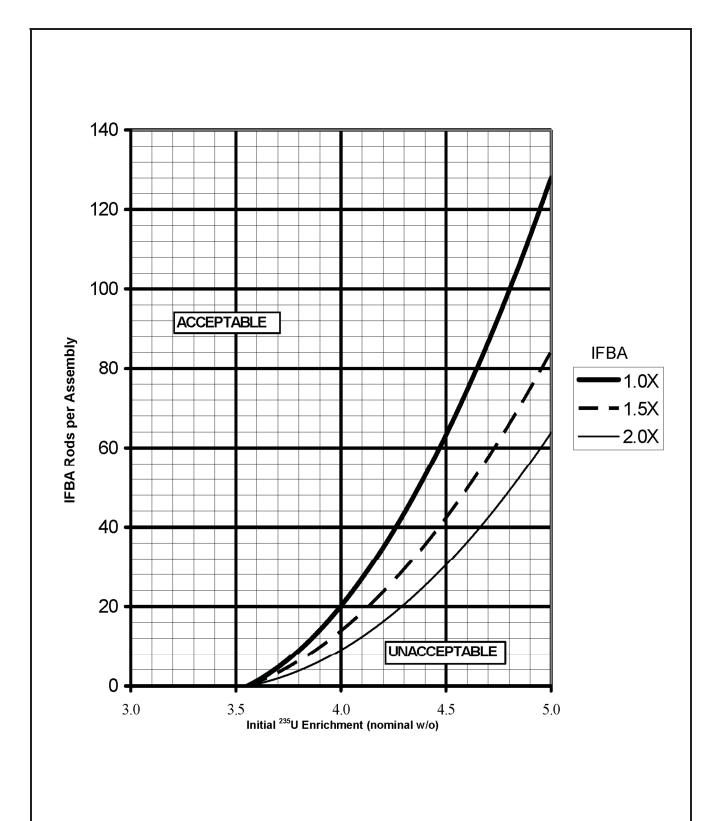
Note:

- 1. A row of empty cells can be used at the interface to separte the configurations.
- 2. It is acceptable to replace an assembly with an empty cell.
- 3. For the 3-Out-of-4 configuration, the row beyond the Low enrichment can swap empty and B assemblies, however the next outer row must change the indicated assembly (*) to an empty cell.
- 4. For the 2-Out-of-4 configuration, the row beyond the Low enrichment can swap empty and B assemblies, however the next outer row of empty and C assemblies must also swap locations.
- 5. If empty cells are in indicated locations (**), then the face adjacent B assemblies can be C assemblies.

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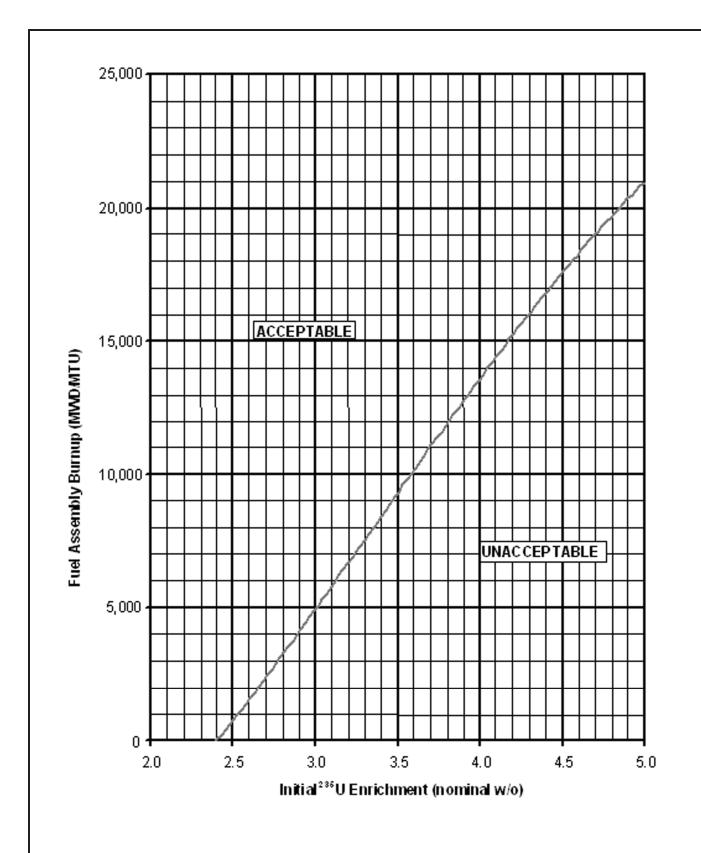
VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 VOGTLE UNIT 2 INTERFACE REQUIREMENTS (3X3 TO EMPTY CELL CHECKERBOARD STORAGE)



REV 13 4/06



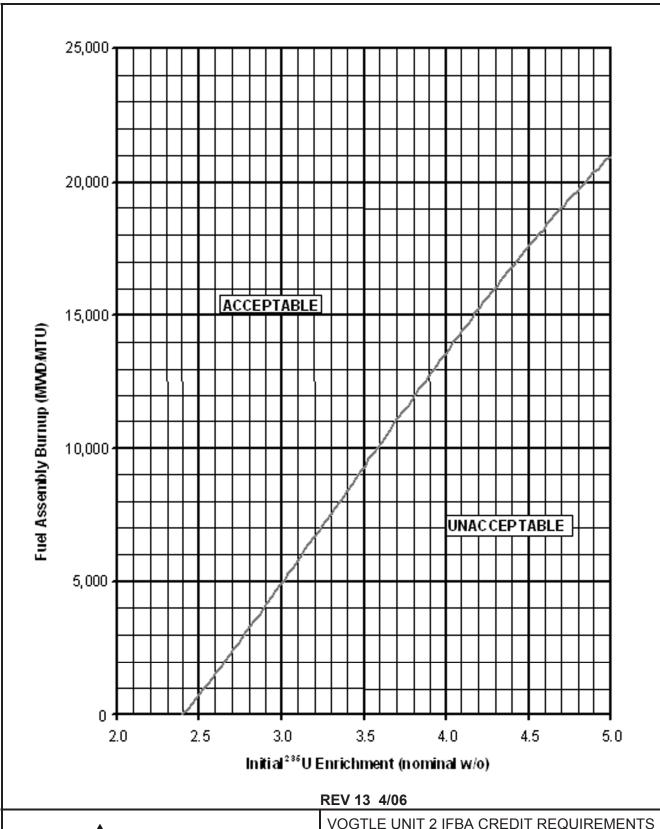
VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 VOGTLE UNIT 1 IFBA CREDIT REQUIREMENTS FOR ALL CELL STORAGE



REV 13 4/06

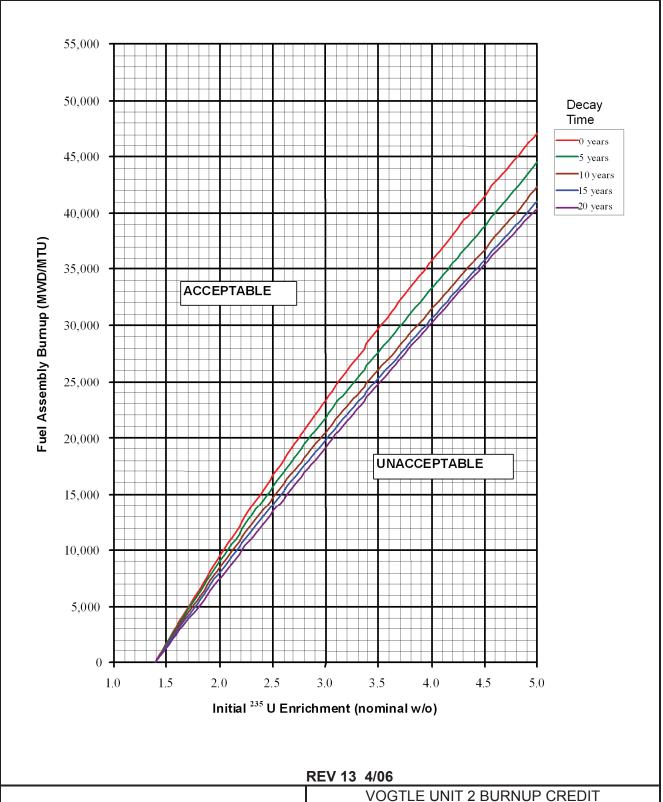


VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 VOGTLE UNIT 2 BURNUP CREDIT REQUIREMENTS FOR 3-OUT-OF-4 STORAGE





VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 VOGTLE UNIT 2 IFBA CREDIT REQUIREMENTS FOR CENTER ASSEMBLY FOR 3X3 STORAGE





VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 VOGTLE UNIT 2 BURNUP CREDIT REQUIREMENTS FOR PERIPHERAL ASSEMBLIES FOR 3X3 STORAGE

4.4 THERMAL AND HYDRAULIC DESIGN

4.4.1 DESIGN BASES

The overall objective of the thermal and hydraulic design of the reactor core is to provide adequate heat transfer compatible with the heat generation distribution in the core so that heat removal by the reactor coolant system (RCS) or the emergency core cooling system (ECCS), when applicable, ensures that the following performances and safety criteria requirements are met:

- A. Fuel damage (defined as penetration of the fission product barrier; i.e., the fuel rod clad) is not expected during normal operation and operational transients (Condition 1) or any transient conditions arising from faults of moderate frequency (Condition 2). It is not possible, however, to preclude a very small number of rod failures. These will be within the capability of the plant cleanup system and are consistent with the plant design bases.
- B. The reactor can be brought to a safe state following a Condition 3 event with only a small fraction of fuel rods damaged (See above definition.), although sufficient fuel damage might occur to preclude resumption of operation without considerable outage time.
- C. The reactor can be brought to a safe state and the core can be kept subcritical with acceptable heat transfer geometry following transients arising from Condition 4 events.

To satisfy the above requirements, the following design bases have been established for the thermal and hydraulic design of the reactor core.

4.4.1.1 <u>Departure from Nucleate Boiling (DNB) Design Basis</u>

4.4.1.1.1 Basis

There will be at least a 95-percent probability that DNB will not occur on the limiting fuel rods during normal operation and operational transients and any transient conditions arising from faults of moderate frequency (Condition 1 and 2 events) at a 95-percent confidence level.

4.4.1.1.2 Discussion

The design method employed to meet the DNB design basis for the VANTAGE + / VANTAGE 5 fuel assemblies is the Revised Thermal Design Procedure (RTDP), reference 81. With the RTDP methodology, uncertainties in plant operating parameters, nuclear and thermal parameters, fuel fabrication parameters, computer codes, and DNB correlation predictions are considered statistically to obtain DNB uncertainty factors. Based on the DNB uncertainty

factors, RTDP design limit DNBR values are determined such that there is at least 95-percent probability at a 95-percent confidence level that DNB will not occur on the most limiting fuel rod during normal operation and operational transients and during transient conditions arising from faults of moderate frequency (Condition I and II events as defined in ANSI N18.2).

Uncertainties in the plant operating parameters (pressurizer pressure, primary coolant temperature, reactor power, and reactor coolant system flow) have been evaluated for the VEGP Units 1 and 2 with RTD bypass loops, reference 82, and for RTD bypass loops eliminated, reference 83. In the DNBR analyses with RTDP, a set of plant operating parameter uncertainties was used to bound operation with either RTD bypass loops or RTD bypass loops eliminated. Only the random portion of the plant operating parameter uncertainties is included in the statistical combination. Any adverse instrumentation bias is treated as either an input to the DNBR calculation or as a direct DNBR penalty. Since the parameter uncertainties are considered in determining the RTDP design limit DNBR values, the plant safety analyses are performed using input parameters at their nominal values.

The RTDP design limit DNBR values are 1.24 and 1.23 for the typical and thimble cells, respectively, for VANTAGE + / VANTAGE 5 fuel.

The design limit DNBR values are used as a basis for the Technical Specifications and are considered in appropriate 10 CFR 50.59 evaluations.

To maintain DNBR margin to offset DNB penalties such as those due to fuel rod bow (paragraph 4.4.2.2.5) and transition core (paragraph 4.4.2.2.6), the safety analyses were performed to DNBR limits higher than the design limit DNBR values. The difference between the design limit DNBRs and the safety analysis limit DNBRs results in available DNBR margin. The net DNBR margin, after consideration of all penalties, is available for operating and design flexibility.

The DNBR analyses for operation at 3626 MWt were based on the continued use of thimble plugging devices. Operation with thimble plugs in place reduces the core bypass flow through the fuel assembly thimble tubes. Bypass flow is assumed to be ineffective for core heat removal. The reduction in core bypass flow for operation with the thimble plugs in place is a DNBR benefit. All of the flow and DNBR values presented in table 4.4-1 are based the use of thimble plugs.

The option of thimble plug removal was included in all of the non-DNBR analyses performed in support of the uprating to 3626 MWt. The allocation of available DNBR margin would be required to support thimble plug removal.

The Standard Thermal Design Procedure (STDP) is used for those analyses where RTDP is not applicable. In the STDP method, the parameters used in analysis are treated in a conservative way from a DNBR standpoint. The parameter uncertainties are applied directly to the plant safety analyses input values to give the lowest minimum DNBR. The DNBR limit for STDP is the appropriate DNB correlation limit increased by sufficient margin to offset the applicable DNBR penalties.

By preventing DNB, adequate heat transfer is ensured between the fuel clad and the reactor coolant, thereby preventing clad damage as a result of inadequate cooling. Maximum fuel rod surface temperature is not a design basis, since it will be within a few degrees of coolant temperature during operation in the nucleate boiling region. Limits provided by the nuclear control and protection systems are such that this design basis will be met for transients

associated with Condition 2 events including overpower transients. There is an additional large DNBR margin at rated power operation and during normal operating transients.

4.4.1.2 <u>Fuel Temperature Design Basis</u>

4.4.1.2.1 Basis

During modes of operation associated with Condition 1 and Condition 2 events, there is at least a 95-percent probability that the peak kW/ft fuel rods will not exceed the UO_2 melting temperature at the 95-percent confidence level. The melting temperature of UO_2 is taken as $5080^{\circ}F$, $^{(1)}$ unirradiated and decreasing $58^{\circ}F$ per 10,000 MWd/tonne of uranium. By precluding UO_2 melting, the fuel geometry is preserved and possible adverse effects of molten UO_2 on the cladding are eliminated. To preclude center melting and as a basis for overpower protection system setpoints, a calculated centerline fuel temperature of $4700^{\circ}F$ has been selected as the overpower limit. This provides sufficient margin for uncertainties in the thermal evaluations as described in paragraph 4.4.2.9.1.

4.4.1.2.2 Discussion

Fuel rod thermal evaluations are performed at rated power, at maximum overpower, and during transients at various burnups. These analyses ensure that this design basis and the fuel integrity design bases given in section 4.2 are met. They also provide input for the evaluation of Condition 3 and 4 events given in chapter 15.

4.4.1.3 Core Flow Design Basis

4.4.1.3.1 Basis

A minimum of 93.6 percent of the thermal flowrate will pass through the fuel rod region of the core and be effective for fuel rod cooling. Coolant flow through the thimble tubes and the leakage from the core barrel-baffle region into the core are not considered effective for heat removal.

4.4.1.3.2 **Discussion**

Core cooling evaluations are based on the thermal flowrate (minimum flow) entering the reactor vessel. A maximum of 6.4 percent of this value is allotted as bypass flow. This includes rod cluster control (RCC) guide thimble cooling flow, head cooling flow, baffle leakage, and leakage to the vessel outlet nozzle.

4.4.1.4 <u>Hydrodynamic Stability Design Basis</u>

Modes of operation associated with Condition 1 and 2 events shall not lead to hydrodynamic instability.

4.4.1.5 Other Considerations

The above design bases together with the fuel clad and fuel assembly design bases given in subsection 4.2.1 are sufficiently comprehensive that additional limits are not required.

Fuel rod diametral gap characteristics, moderator-coolant flow velocity and distribution, and moderator void are not inherently limiting. Each of these parameters is incorporated into the thermal and hydraulic models used to ensure the above-mentioned design criteria are met. For instance, the fuel rod diametral gap characteristics change with time (paragraph 4.2.3.3), and the fuel rod integrity is evaluated on that basis. The effect of the moderator flow velocity and distribution (paragraph 4.4.2.2) and moderator void distribution (paragraph 4.4.2.4) are included in the core thermal evaluation and thus affect the design bases.

Meeting the fuel clad integrity criteria covers possible effects of clad temperature limitations. As noted in paragraph 4.2.3.3, the fuel rod conditions change with time. A single clad temperature limit for Condition 1 or Condition 2 events is not appropriate, since it would of necessity be overly conservative. A clad temperature limit is applied to the loss-of-coolant accident (LOCA) (subsection 15.6.5), control rod ejection accident (subsection 15.4.8), and locked rotor accident (subsection 15.3.3).

4.4.2 DESCRIPTION OF THERMAL AND HYDRAULIC DESIGN OF THE REACTOR CORE

4.4.2.1 Summary Comparison

Table 4.4-1 provides the design parameters at 3626 MWt for the 17 x 17 VANTAGE + / VANTAGE 5 fuel. The LOPAR fuel is not analyzed for use at 3626 MWt. The LOPAR design parameters at 3565 MWt are retained in table 4.4-1 for historical purposes.

4.4.2.2 Critical Heat Flux Ratio or DNBR and Mixing Technology

The minimum DNBRs for the rated power, design overpower, and anticipated transient conditions are given in table 4.4-1. The minimum DNBR in the limiting flow channel is typically downstream of the peak heat flux location (hotspot) due to the increased downstream enthalpy rise.

DNBRs are calculated by using the correlation and definitions described in paragraphs 4.4.2.2.1 and 4.4.2.2.2. The VIPRE-01 computer code (discussed in paragraph 4.4.4.5) is used to determine the flow distribution in the core and the local conditions in the hot channel for use in the DNB correlation. The use of hot channel factors is discussed in paragraphs 4.4.4.3.1 (nuclear hot channel factors) and 4.4.2.2.4 (engineering hot channel factors).

4.4.2.2.1 DNB Technology

The primary DNB correlation that was used for the analysis of the 17 x 17 LOPAR fuel was the WRB-1 correlation (reference 84).

The WRB-1 correlation was developed based exclusively on the large bank of mixing vane grid rod bundle CHF data (over 1100 points) that Westinghouse has collected. The WRB-1 correlation, based on local fluid conditions, represents the rod bundle data with better accuracy over a wide range of variables than the previous correlation used in design. This correlation accounts directly for both typical and thimble cold wall cell effects, uniform and nonuniform heat flux profiles, and variations in rod heated length and in grid spacing.

The applicable range of parameters for the WRB-1 correlation is:

Pressure $1440 \le P \le 2490 \text{ psia}$ Local mass velocity $0.9 \le G_{loc}/10^6 \le \\ 3.7 \text{ lb/ft}^2\text{-hr}$

 $\begin{tabular}{lll} Local quality & -0.2 \le X_{loc} \le 0.3 \\ Heated length, inlet to CHF location & L_h \le 14 feet \\ \end{tabular}$

 $\begin{array}{ll} \text{Grid spacing} & 13 \leq g_{sp} \leq 32 \text{ inches} \\ \text{Equivalent hydraulic diameter} & 0.37 \leq d_e \leq 0.60 \text{ inches} \\ \text{Equivalent heated hydraulic diameter} & 0.46 \leq d_h \leq 0.59 \text{ inches} \\ \end{array}$

Figure 4.4-1 shows measured critical heat flux plotted against predicted critical heat flux using the WRB-1 correlation.

A correlation limit DNBR of 1.17 for the WRB-1 correlation has been approved by the NRC for 17 x 17 LOPAR fuel.

The primary DNB correlation used for the analysis of the VANTAGE + / VANTAGE 5 fuel is the WRB-2 correlation (reference 85). The WRB-2 DNB correlation was developed to take credit for the VANTAGE 5 intermediate flow mixer (IFM) grid design. A limit of 1.17 is applicable for the WRB-2 correlation. Figure 4.4-1 shows measured critical heat flux plotted against predicted critical heat flux using the WRB-2 correlation.

Use of this correlation has been conservatively modified to utilize a penalty above a certain high quality threshold within the approved ranges (reference 102).

The applicable range of parameters for the WRB-2 correlation is:

Pressure $1440 \le P \le 2490 \text{ psia}$

Local mass velocity $0.9 \le G_{loc}/10^6 \le 3.7 \text{ lb/ft}^2\text{-h}$

Local quality $-0.1 \le X_{loc} \le 0.3$

Heated length, inlet to CHF location $L_h \le 14 \text{ ft}$

Grid spacing $10 \le g_{sp} \le 26$ in.

Equivalent hydraulic diameter $0.33 \leq d_e \leq 0.5101 \text{ in}.$

Equivalent heated hydraulic diameter $0.45 \le d_h \le 0.66$ in.

The W-3 DNB correlation, references 86 and 4, is used for both fuel types where the primary DNBR correlations are not applicable. The WRB-1 and WRB-2 correlations were developed based on mixing vane data and therefore are only applicable in the heated rod spans above the first mixing vane grid. The W-3 correlation, which does not take credit for mixing vane grids, is used to calculate DNBR values in the heated region below the first mixing vane grid. In addition, the W-3 correlation is applied in the analysis of accident conditions where the system pressure is below the range of the primary correlations. For system pressures in the range of 500 to 1000 psia, the W-3 correlations limit is 1.45, reference 87. For system pressures greater than 1000 psia, the W-3 correlation limit is 1.30. A cold wall factor, reference 88, is applied to the W-3 DNB correlation to account for the presence of the unheated thimble surfaces.

4.4.2.2.2 Definition of DNBR

The DNB heat flux ratio (DNBR) as applied to typical cells (flow cells with all walls heated) and thimble cells (flow cells with heated and unheated walls) is defined as:

$$DNBR = \frac{q''_{DNB,N}}{q''_{loc}}$$
 (1)

where:

$$q_{DNB,N}'' = \frac{q_{DNB,EU}''}{F} \tag{2}$$

 $q''_{DNB,EU} =$ the uniform DNB heat flux as predicted by the WRB -1 DNB correlation, the WRB - 2 DNB correlation, or the W - 3 DNB correlation (typical cell only).

F = the flux shape factor to account for nonuniform axial heat flux distributions⁽⁸⁾ with the term "C" modified as in reference 4.

 q''_{loc} = the actual local heat flux.

The DNBR as applied to the W-3 DNB correlation when a cold wall (CW) is present is:

$$DNBR = \frac{q_{DNB,N,CW}^{\prime\prime}}{q_{loc}^{\prime\prime}}$$
 (3)

where:

$$q_{DNB,N,CW}'' = \frac{q_{DNB,EU,Dh}' \times CWF}{F}$$
(4)

where:

q"_{DNB,EU,Dh} = the uniform DNB heat flux as predicted by the W - 3 cold wall DNB correlation(4) when not all flow cell walls are heated (thimble CW cell).

CW factor⁽⁴⁾ =
$$1.0 - \text{Ru} \ 13.76 - 1.372e^{1.78x} - 4.732$$

$$\left(\frac{\text{G}}{10^6}\right)^{-0.0535} - 0.0619 \left(\frac{\text{P}}{1000}\right)^{0.14} - 8.509 \,\text{Dh}^{0.107}$$
 Ru = $1.0 - \text{De/Dh}$

4.4.2.2.3 Mixing Technology

The rate of heat exchange by mixing between flow channels is proportional to the difference in the local mean fluid enthalpy of the respective channels, the local fluid density, and the flow velocity. The proportionality is expressed by the dimensionless thermal diffusion coefficient (TDC) which is defined as:

$$TDC = \frac{w'}{\rho Va}$$
 (5)

where:

w' = flow exchange rate per unit length (lbm/ft-s).

 ρ = fluid density (lbm/ft³).

V = fluid velocity (ft/s).

Pressure

a = lateral flow area between channels per unit length (ft^2/ft).

The application of the TDC in the THINC analysis for determining the overall mixing effect or heat exchange rate is presented in reference 10. The application of the TDC in the VIPRE-01 analysis is presented in reference 101.

Various mixing tests have been performed at Columbia University. (9) These series of tests, using the "R" mixing vane grid design on 13-, 26-, and 32-in. grid spacing, were conducted in pressurized water loops at Reynolds numbers similar to that of a pressurized water reactor (PWR) core under the following single- and two-phase (subcooled boiling) flow conditions:

1500 to 2400 psia

•	Inlet temperature	332 to 642°F
•	Mass velocity	1.0 to 3.5 x 10^6 lbm/h-ft ²

• Reynolds number 1.34 to 7.45 x 10⁵

Bulk outlet quality -52.1 to -13.5 percent

TDC is determined by comparing the THINC code predictions with the measured subchannel exit temperatures. Data for 26-in. axial grid spacing are presented in figure 4.4-2, where the TDC coefficient is plotted versus the Reynolds number. TDC is found to be independent of the Reynolds number, mass velocity, pressure, and quality over the ranges tested. The two-phase data (local, subcooled boiling) fell within the scatter of the single-phase data. The effect of two-

phase flow on the value of TDC has been demonstrated by Cadek, (9) Rowe and Angle, (11)(12) and Gonzalez-Santalo and Griffith. (13) In the subcooled boiling region, the values of TDC were indistinguishable from the single-phase values. In the quality region, Rowe and Angle show that in the case with rod spacing similar to that in PWR core geometry, the value of TDC increased with quality to a point and then decreased, but never below the single-phase value. Gonzalez-Santalo and Griffith show that the mixing coefficient increased as the void fraction increased.

The data from these tests on the R-grid showed that a design TDC value of 0.038 (for 26-in. grid spacing) can be used in determining the effect of coolant mixing in the THINC or VIPRE-01 analysis. A mixing test program similar to the one described above was conducted at Columbia University for the current 17 x 17 geometry and mixing vane grids on 26-in. spacing. (14) The mean value of TDC obtained from these tests was 0.059, and all data were well above the current design value of 0.038.

Since the actual grid spacing for 17 x 17 LOPAR fuel is approximately 20 in., additional margin is available for this design, as the value of TDC increases as grid spacing decreases. (9)

The inclusion of three intermediate flow mixer grids in the upper span of the VANTAGE 5 fuel assembly results in a grid spacing of approximately 10 inches. Per Reference 85, a TDC value of 0.038 was chosen as a conservatively low value for use in VANTAGE 5 to determine the effect of coolant mixing in the core thermal performance analysis.

4.4.2.2.4 Hot Channel Factors

The total hot channel factors for heat flux and enthalpy rise are defined as the maximum-to-core average ratios of these quantities. The heat flux hot channel factor considers the local maximum linear heat generation rate at a point (the hotspot), and the enthalpy rise hot channel factor involves the maximum integrated value along a channel (the hot channel).

Each of the total hot channel factors is composed of a nuclear hot channel factor (paragraph 4.4.4.3) describing the neutron power distribution and an engineering hot channel factor, which allows for variations in flow conditions and fabrication tolerances. The engineering hot channel factors are made up of subfactors which account for the influence of the variations of fuel pellet diameter, density, enrichment, and eccentricity; inlet flow distribution; flow redistribution; and flow mixing.

A. Heat Flux Engineering Hot Channel Factor, FQ

The heat flux engineering hot channel factor is used to evaluate the maximum linear heat generation rate in the core. This subfactor is determined by statistically combining the fabrication variations for fuel pellet diameter, density, and enrichment and has a value of 1.03 at the 95-percent probability level with 95-percent confidence. As shown in reference 15, no DNB penalty need be taken for the short, relatively low-intensity heat flux spikes caused by variations in the above parameters, as well as fuel pellet eccentricity and fuel rod diameter variation.

B. Enthalpy Rise Engineering Hot Channel Factor, $F_{\Delta H}^{E}$

The effect of variations in flow conditions and fabrication tolerances on the hot channel enthalpy rise is directly considered in the VIPRE-01 subchannel analysis

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(paragraph 4.4.4.5) under any reactor operating condition. The items considered contributing to the enthalpy rise engineering hot channel factor are discussed below:

1. Pellet Diameter, Density, and Enrichment

Variations in pellet diameter, density, and enrichment are considered statistically in establishing the limit DNBRs (paragraph 4.4.1.1.2) for the Revised Thermal Design Procedure (reference 81) employed in this application. Uncertainties in these variables are determined from sampling of manufacturing data.

2. Inlet Flow Maldistribution

The consideration of inlet flow maldistribution in core thermal performances is discussed in paragraph 4.4.4.2.2. A design basis of 5-percent reduction in coolant flow to the hot assembly is used in the VIPRE-01 analysis.

3. Flow Redistribution

The flow redistribution accounts for the reduction in flow in the hot channel resulting from the high flow resistance in the channel due to the local or bulk boiling. The effect of the nonuniform power distribution is inherently considered in the VIPRE-01 analysis for every operating condition evaluated.

4. Flow Mixing

The subchannel mixing model incorporated in the VIPRE-01 code and used in reactor design is based on experimental data⁽¹⁶⁾ discussed in paragraphs 4.4.2.2.3 and 4.4.4.5. The mixing vanes incorporated in the spacer grid design induce additional flow mixing between the various flow channels in a fuel assembly as well as between adjacent assemblies. This mixing reduces the enthalpy rise in the hot channel resulting from local power peaking or unfavorable mechanical tolerances.

4.4.2.2.5 Effects of Rod Bow on DNBR

The phenomenon of fuel rod bowing, as described in reference 79, must be accounted for in the DNBR safety analysis of Condition I and Condition II events for each plant application. Applicable generic credits for margin resulting from retained conservatism in the evaluation of DNBR and/or margin obtained from measured plant operating parameters (such as $F_{\Delta H}^{N}$ or core flow), which are less limiting than those required by the plant safety analysis, can be used to offset the effect of rod bow.

For the safety analysis of the VEGP units, sufficient DNBR margin was maintained (paragraph 4.4.1.1.2) to accommodate full and low flow rod bow DNBR penalties which are based on the methodology in reference 80. The rod bow DNBR penalties that are applicable to LOPAR fuel assembly analyses using the WRB-1 DNB correlation and to VANTAGE 5 fuel assembly analyses using the WRB-2 DNB correlation were determined using the methodology in reference 80.

The maximum rod bow penalties (< 2 percent DNBR) accounted for in the design safety analysis are based on an assembly average burnup of 24,000 MWd/Mtu. At burnups greater than 24,000 MWd/Mtu, credit is taken for the effect of F_{AH}^{N} burndown, due to the decrease in

fissionable isotopes and the buildup of fission product inventory, and no additional rod bow penalty is required (reference 95).

In the upper spans of the VANTAGE 5 fuel assembly, additional restraint is provided with the intermediate flow mixer grids such that the grid-to-grid spacing in those spans with IFM grids is approximately 10 inches compared to approximately 20 inches in the other spans. Using the NRC approved scaling factor results in predicted channel closure in the limiting 10-inch spans of less than 50-percent closure. Therefore, no rod bow DNBR penalty is required in the 10-inch spans in the VANTAGE 5 safety analyses.

4.4.2.2.6 Transition Core DNB Methodology

The LOPAR and VANTAGE 5 designs have been shown to be hydraulically compatible in reference 85.

The Westinghouse transition core DNB methodology is given in references 89, 90, and 91. Using this methodology, transition cores are analyzed as if the entire core consisted of one assembly type (full LOPAR or full VANTAGE 5). The resultant DNBRs are then reduced by the appropriate transition core penalty.

The VANTAGE 5 fuel assembly has a higher mixing vane grid loss coefficient relative to the LOPAR mixing vane grid loss coefficient. In addition, the VANTAGE 5 fuel assembly has IFM grids located in spans between mixing vane grids, where no grid exists in the LOPAR assembly. The higher loss coefficients and the additional grids introduce localized flow redistribution from the VANTAGE 5 fuel assembly into the LOPAR assembly at the axial zones near the mixing vane grid and the IFM grid position in a transition core. Between the grids, the tendency for velocity equalization in parallel open channels causes flow to return to the VANTAGE 5 fuel assembly. The localized flow redistribution described above actually benefits the LOPAR assembly. This benefit more than offsets the slight mass flow bias due to velocity equalization at nongridded locations. Thus, the analysis for a full core of LOPAR is appropriate for that fuel type in a transition core. There is no transition core DNBR penalty for the LOPAR fuel.

The transition core DNBR penalty for VANTAGE 5 fuel is discussed in references 92 and 93. The transition core penalty is a function of the number of VANTAGE 5 fuel assemblies in the core, reference 94. Sufficient DNBR margin is maintained in the VANTAGE 5 safety analysis to completely offset this transition core penalty.

4.4.2.3 Linear Heat Generation Rate

The core average and maximum linear heat generation rates are given in table 4.4-1. The method of determining the maximum linear heat generation rate is given in paragraph 4.3.2.2.

4.4.2.4 Void Fraction Distribution

The VIPRE-01 calculated core average and the hot subchannel maximum and average void fractions are presented in table 4.4-2 for operation at full power. The void models used in the VIPRE-01 code are described in paragraph 4.4.2.7.3.

4.4.2.5 Core Coolant Flow Distribution

Assembly average coolant mass velocity and enthalpy at various radial and axial core locations are given in figures 4.4-3 through 4.4-5. Typical coolant enthalpy rise and flow distributions for the 4-ft elevation (one-third of core height) are shown in figure 4.4-3, for the 8-ft elevation (two-thirds of core height) in figure 4.4-4, and at the core exit in figure 4.4-5. These distributions are representative of a Westinghouse four-loop plant. The THINC code analysis for this case utilized a uniform code inlet enthalpy and inlet flow distribution. No orificing is employed in the reactor design.

4.4.2.6 Core Pressure Drops and Hydraulic Loads

4.4.2.6.1 Core Pressure Drops

The analytical model and experimental data used to calculate the pressure drops shown in table 4.4-1 are described in paragraph 4.4.2.7. The core pressure drop includes the fuel assembly, lower core plate, and upper core plate pressure drops. The full-power operation pressure drop values shown in table 4.4-1 are the unrecoverable pressure drops across the vessel, including the inlet and outlet nozzles, and across the core. These pressure drops are based on the best-estimate flow for actual plant operating conditions as described in subsection 5.1.4. This subsection also defines and describes the thermal design flow (minimum flow), which is the basis for reactor core thermal performance and the mechanical design flow (maximum flow), which is used in the mechanical design of the reactor vessel internals and fuel assemblies. Since the best-estimate flow is that flow which is most likely to exist in an operating plant, the calculated core pressure drops in table 4.4-1 are based on this best-estimate flow rather than the thermal design flow.

Uncertainties associated with the core pressure drop values are discussed in paragraph 4.4.2.9.2.

4.4.2.6.2 Hydraulic Loads

The fuel assembly holddown springs (figure 4.2-2) are designed to keep the fuel assemblies in contact with the lower core plate under all Condition 1 and 2 events except the turbine overspeed transient associated with a loss of external load. The holddown springs are designed to tolerate the possibility of an overdeflection associated with fuel assembly liftoff for this case and provide contact between the fuel assembly and the lower core plate following this transient. More adverse flow conditions occur during a LOCA. These conditions are presented in subsection 15.6.5.

Hydraulic loads at normal operating conditions are calculated considering the mechanical design flow, which is described in section 5.1, and accounting for the minimum core bypass flow based on manufacturing tolerances. Core hydraulic loads at cold plant startup conditions are based on the cold mechanical design flow, but are adjusted to account for the coolant density difference. Conservative core hydraulic loads for a pump overspeed transient, which could possibly create flowrate 20 percent greater than the mechanical design flow, are evaluated to be approximately twice the fuel assembly weight.

The hydraulic verification tests for the LOPAR fuel assembly and the VANTAGE 5 fuel assembly are discussed in references 18 and 85, respectively.

4.4.2.7 Correlation and Physical Data

4.4.2.7.1 Surface Heat Transfer Coefficients

Forced convection heat transfer coefficients are obtained from the familiar Dittus-Boelter correlation, (19) with the properties evaluated at bulk fluid conditions:

$$\frac{hD_e}{K} = 0.023 \frac{D_e G^{0.8}}{\mu} \frac{C_{\rho} \mu^{0.4}}{K}$$
 (6)

where:

h = heat transfer coefficient (Btu/h-ft 2 - $^\circ$ F).

D_e = equivalent diameter (ft).

K = thermal conductivity (Btu/h-ft-°F).

 $G = mass velocity (lbm/h-ft^2).$

 μ = dynamic viscosity (lbm/ft-h).

 $C\rho$ = heat capacity (Btu/lb-°F).

This correlation has been shown to be conservative⁽²⁰⁾ for rod bundle geometries with pitch-to-diameter ratios in the range used by PWRs.

The onset of nucleate boiling occurs when the clad wall temperature reaches the amount of superheat predicted by Thom's correlation. (21) After this occurrence the outer clad wall temperature is determined by:

$$\Delta T_{\text{sat}} = [0.072 \exp (-P/1260)] (q'')^{0.5}$$
 (7)

where:

 ΔT_{sat} = wall superheat, $T_w - T_{sat}$ (°F).

 $q'' = wall heat flux (Btu/h-ft^2).$

P = pressure (psia).

 T_w = outer clad wall temperature (°F).

 T_{sat} = saturation temperature of coolant at P (°F).

4.4.2.7.2 Total Core and Vessel Pressure Drop

Unrecoverable pressure losses occur as a result of viscous drag (friction) and/or geometry changes (form) in the fluid flow path. The flow field is assumed to be incompressible, turbulent, single-phase water. These assumptions apply to the core and vessel pressure drop calculations for the purpose of establishing the primary loop flowrate. Two-phase considerations are neglected in the vessel pressure drop evaluation because the core average void is negligible (table 4.4-2). Two-phase flow considerations in the core thermal subchannel analyses are considered and the models are discussed in paragraph 4.4.4.2.3. Core and vessel pressure losses are calculated by equations of the form:

$$\Delta \rho_{\rm L} = \left(K + F \frac{L}{D_{\rm e}} \right) \frac{\rho V^2}{2g_c(144)} \tag{8}$$

where:

 $\Delta \rho_L$ = unrecoverable pressure drop (lb/in.²).

 ρ = fluid density (lbm/ft³).

L = length (ft).

De = equivalent diameter (ft).

V = fluid velocity (ft/s).

 $g_c = 32.174 \text{ (lbm-ft/lb}_f - s^2).$

K = form loss coefficient (dimensionless).

F = friction loss coefficient (dimensionless).

Fluid density is assumed to be constant at the appropriate value for each component in the core and vessel. Because of the complex core and vessel flow geometry, precise analytical values for the form and friction loss coefficients are not available. Therefore, experimental values for these coefficients are obtained from geometrically similar models.

Values are quoted in table 4.4-1 for unrecoverable pressure loss across the reactor vessel, including the inlet and outlet nozzles, and across the core. The results of full-scale tests of core components and fuel assemblies were utilized in developing the core pressure loss characteristic. The pressure drop for the vessel was obtained by combining the core loss with correlation of one-seventh scale model hydraulic test data on a number of vessels⁽²²⁾⁽²³⁾ and form loss relationships.⁽²⁴⁾ Moody⁽²⁵⁾ curves were used to obtain the single-phase friction factors.

Tests of the primary coolant loop flowrates will be made (paragraph 4.4.5.1) prior to initial criticality to verify that the flowrates used in the design, which were determined in part from the pressure losses calculated by the method described here, are conservative.

4.4.2.7.3 Void Fraction Correlation

VIPRE-01 considers two-phase flow in two steps. First, a quality model is used to compute the flowing vapor mass fraction (true quality) including the effects of subcooled boiling. Then, given the true quality, a bulk void model is applied to compute the vapor volume fraction (void fraction).

VIPRE-01 uses a profile fit model (100) for determining subcooled quality. It calculates the local vapor volumetric fraction in forced convection boiling by: 1) predicting the point of bubble departure from the heated surface, and 2) postulating a relationship between the true local vapor fraction and the corresponding thermal equilibrium value.

The void fraction in the bulk boiling region is predicted by using homogeneous flow theory and assuming no slip. The void fraction in this region is therefore a function only of the thermodynamic quality.

4.4.2.8 <u>Thermal Effects of Operational Transients</u>

DNB core safety limits are generated as a function of coolant temperature, pressure, core power, and axial power imbalance. Steady-state operation within these safety limits ensures that the DNBR design basis is met. Figure 15.0.6-1 shows the DNBR limit lines and the resulting overtemperature ΔT trip lines (which become part of the Technical Specifications), plotted as ΔT versus T_{avg} for various pressures. This system provides adequate protection against anticipated operational transients that are slow with respect to fluid transport delays in the primary system. In addition, for fast transients (e.g., uncontrolled rod bank withdrawal at power incident (subsection 15.4.2)), specific protection functions are provided as described in section 7.2. The use of these protection functions is described in chapter 15.

4.4.2.9 Uncertainties in Estimates

4.4.2.9.1 Uncertainties in Fuel and Clad Temperatures

As discussed in paragraph 4.4.2.11, the fuel temperature is a function of crud, oxide, clad, pellet-clad gap, and pellet conductances. Uncertainties in the fuel temperature calculation are essentially of two types: fabrication uncertainties, such as variations in the pellet and clad dimensions and the pellet density; and model uncertainties, such as variations in the pellet conductivity and the gap conductance. These uncertainties have been quantified by comparison of the thermal model to the in-pile thermocouple measurements, (29-35) by out-of-pile measurements of the fuel and clad properties, and by measurements of the fuel and clad dimensions during fabrication. The resulting uncertainties are then used in all evaluations involving the fuel temperature. The effect of densification on fuel temperature uncertainties is also included in the calculation of the total uncertainty.

In addition to the temperature uncertainty described above, the measurement uncertainty in determining the local power and the effect of density and enrichment variations on the local power are considered in establishing the heat flux hot channel factor. These uncertainties are described in paragraph 4.3.2.2.1.

Reactor trip setpoints, as specified in the Technical Specifications, include allowance for instrument and measurement uncertainties such as calorimetric error, instrument drift and channel reproducibility, temperature measurement uncertainties, noise, and heat capacity variations.

Uncertainty in determining the cladding temperature results from uncertainties in the crud and oxide thicknesses. Because of the excellent heat transfer between the surface of the rod and the coolant, the film temperature drop does not appreciably contribute to the uncertainty.

4.4.2.9.2 Uncertainties in Pressure Drops

Core and vessel pressure drops based on the best-estimate flow, as described in section 5.1, are quoted in table 4.4-1. The uncertainties quoted are based on the uncertainties in both the test results and the analytical extension of these values to the reactor application.

A major use of the core and vessel pressure drops is to determine the primary system coolant flowrates, as discussed in section 5.1. In addition, as discussed in paragraph 4.4.5.1, tests on the primary system prior to initial criticality will be made to verify that a conservative primary system coolant flowrate has been used in the design and analyses of the plant.

4.4.2.9.3 Uncertainties Due to Inlet Flow Maldistribution

The effects of uncertainties in the inlet flow maldistribution criteria used in the core thermal analyses are discussed in paragraph 4.4.4.2.2.

4.4.2.9.4 Uncertainty in DNB Correlation

The uncertainty in the DNB correlation (paragraph 4.4.2.2) can be written as a statement on the probability of not being in DNB based on the statistics of the DNB data. This is discussed in paragraph 4.4.2.2.2.

4.4.2.9.5 Uncertainties in DNBR Calculations

The uncertainties in the DNBRs calculated by VIPRE-01 analysis (paragraph 4.4.4.5) due to uncertainties in the nuclear peaking factors are accounted for by applying conservatively high values of the nuclear peaking factors and including measurement error allowances in the statistical evaluation of the limit DNBR (paragraph 4.4.1.1) using the Revised Thermal Design Procedure (reference 81). In addition, conservative values for the engineering hot channel factors are used as discussed in paragraph 4.4.2.2.4. The results of a sensitivity study⁽¹⁷⁾ with THINC-IV show that the minimum DNBR in the hot channel is relatively insensitive to variations in the core-wide radial power distribution (for the same value of $F_{\Delta H}^{N}$). VIPRE-01 was demonstrated to be equivalent to THINC-IV in reference 101.

The ability of the VIPRE computer code to accurately predict flow and enthalpy distributions in rod bundles is discussed in paragraph 4.4.4.5 and in reference101. Studies (100, 101) have been performed to determine the sensitivity of the minimum DNBR in the hot channel to void fraction

correlation (paragraph 4.4.2.7.3) and the inlet flow distributions. The results of these studies show that the minimum DNBR is relatively insensitive to variation in these parameters. Furthermore, the VIPRE flow field model for predicting conditions in the hot channels is consistent with that used in the derivation of the DNB correlation limits, including void/quality modeling, turbulent mixing and crossflow, and two-phase friction (101).

4.4.2.9.6 Uncertainties in Flowrates

The uncertainties associated with loop flowrates are discussed in section 5.1. A thermal design flow is defined for use in core thermal performance evaluations which accounts for both prediction and measurement uncertainties. In addition, another 6.4 percent of the thermal design flow is assumed to be ineffective for core heat removal capability because it bypasses the core through the various available vessel flow paths described in paragraph 4.4.4.2.1.

4.4.2.9.7 Uncertainties in Hydraulic Loads

As discussed in section 4.4.2.6.2, hydraulic loads on the fuel assembly are evaluated for a pump overspeed transient which creates flowrates 20-percent greater than the mechanical design flow. As stated in section 5.1, the mechanical design flow is greater than the best estimate or most likely flowrate value for the actual plant operating condition.

4.4.2.9.8 Uncertainty in Mixing Coefficient

The value of the mixing coefficient, TDC, used in VIPRE-01 analyses for this application is 0.038 for LOPAR fuel and VANTAGE 5 fuel.

The results of the mixing tests done on 17 x 17 LOPAR geometry, as discussed in paragraph 4.4.2.2.3, had a mean value of TDC of 0.059 and standard deviation of s equal to 0.007. Hence, the current design value of TDC is almost three standard deviations below the mean for 26-in. grid spacing.

4.4.2.10 Flux Tilt Considerations

Significant quadrant power tilts are not anticipated during normal operation since this phenomenon is caused by some asymmetric perturbation. A dropped or misaligned rod cluster control assembly (RCCA) could cause changes in hot channel factors; however, these events are analyzed separately in chapter 15.

Other possible causes for quadrant power tilts include X-Y xenon transients, inlet temperature mismatches, enrichment variations within tolerances, and so forth.

In addition to unanticipated quadrant power tilts as described above, other readily explainable asymmetries may be observed during calibration of the excore detector quadrant power tilt alarm. During operation, incore maps are taken at least one per month and additional maps are obtained periodically for calibration purposes. Each of these maps is reviewed for deviations from the expected power distributions. Asymmetry in the core, from quadrant to quadrant, is

frequently a consequence of the design when assembly and/or component shuffling and rotation requirements do not allow exact symmetry preservation. In each case, the acceptability of an observed asymmetry, planned or otherwise, depends solely on meeting the required accident analyses assumptions. In practice, once acceptability has been established by review of the incore maps, the quadrant power tilt alarms and related instrumentation are adjusted to indicate zero quadrant power tilt ratio as the final step in the calibration process. This action ensures that the instrumentation is correctly calibrated to alarm in the event an unexplained or unanticipated change occurs in the quadrant-to-quadrant relationships between calibration intervals. Proper functioning of the quadrant power tilt alarm is significant; no allowances are made in the design for increased hot channel factors due to unexpected developing flux tilts, since all likely causes are prevented by design or procedures or are specifically analyzed. Finally, in the event that unexplained flux tilts do occur, the Technical Specifications provide appropriate corrective actions to ensure continued safe operation of the reactor.

4.4.2.11 <u>Fuel and Cladding Temperatures</u>

Consistent with the thermal-hydraulic design bases described in subsection 4.4.1, the following discussion pertains mainly to fuel pellet temperature evaluation. A discussion of fuel clad integrity is presented in paragraph 4.2.3.1.

The thermal-hydraulic design ensures that the maximum fuel temperature is below the melting point of UO_2 (paragraph 4.4.1.2). To preclude center melting and as a basis for overpower protection system setpoints, a calculated centerline fuel temperature of $4700^{\circ}F$ has been selected as the overpower limit. This provides sufficient margin for uncertainties in the thermal evaluations as described in paragraph 4.4.2.9.1. The temperature distribution within the fuel pellet is predominantly a function of the local power density and the UO_2 thermal conductivity. However, the computation of radial fuel temperature distributions combines crud, oxide, clad gap, and pellet conductances. The factors which influence these conductances, such as gap size (or contact pressure), internal gas pressure, gas composition, pellet density, and radial power distribution within the pellet, etc., have been combined into a semi empirical thermal model (paragraph 4.2.3.3) which includes a model for time-dependent fuel densification as given in references 96 and 99. This thermal model enables the determination of these factors and their net effects on temperature profiles. The temperature predictions have been compared to in-pile fuel temperature measurements (29-35, 97) and melt radius data (49)(50) with good results.

Fuel rod thermal evaluations (fuel centerline, average and surface temperatures) are performed several times in the fuel rod lifetime (with consideration of time-dependent densification) to determine the maximum fuel temperatures.

The principal factors employed in the determination of the fuel temperature are discussed below.

4.4.2.11.1 UO₂ Thermal Conductivity

The thermal conductivity of uranium dioxide was evaluated from data reported by Howard, <u>et al.</u>;⁽³⁶⁾ Lucks, <u>et al.</u>;⁽³⁷⁾ Daniel, <u>et al.</u>;⁽³⁸⁾ Feith;⁽³⁹⁾ Vogt, <u>et al.</u>;⁽⁴⁰⁾ Nishijima, <u>et al.</u>;⁽⁴¹⁾ Wheeler, <u>et al.</u>;⁽⁴²⁾ Godfrey, <u>et al.</u>;⁽⁴³⁾ Stora, <u>et al.</u>;⁽⁴⁴⁾ Bush;⁽⁴⁵⁾ Asamoto, <u>et al.</u>;⁽⁴⁶⁾ Kruger;⁽⁴⁷⁾ and Gyllander.⁽⁵¹⁾

At the higher temperatures, thermal conductivity is best obtained by utilizing the integral conductivity to melt which can be determined with more certainty. From an examination of the data, it has been concluded that the best estimate is:

$$\int_{0}^{2800} Kdt = 93 W/cm$$

This conclusion is based on the integral values reported by Gyllander; ⁽⁵¹⁾ Lyons, <u>et al.</u>; ⁽⁵²⁾ Copolin, <u>et al.</u>; Duncan; ⁽⁴⁹⁾ Bain; and Stora.

The design curve for the thermal conductivity is shown in figure 4.4-6. The section of the curve at temperatures between 0°C and 1300°C is in excellent agreement with the recommendation of the International Atomic Energy Agency (IAEA) panel. The section of the curve above 1300°C is derived for an integral value of 93 W/cm. (49)(51)(55)

Thermal conductivity for UO₂ at 95-percent theoretical density can be represented best by the following equation:

$$K = \frac{1}{11.8 + 0.0238 \,\mathrm{T}} + 8.775 \times 10^{-13} \,\mathrm{T}^3 \tag{9}$$

where:

 $K = W/cm^{\circ}C$.

 $T = {}^{\circ}C$

4.4.2.11.2 Radial Power Distribution in UO₂ Fuel Rods

An accurate description of the radial power distribution as a function of burnup is needed for determining the power level for incipient fuel melting and other important performance parameters, such as pellet thermal expansion, fuel swelling, and fission gas release rates. Radial power distribution in UO_2 fuel rods is determined with the neutron transport theory code, LASER. The LASER code has been validated by comparing the code predictions on radial burnup and isotopic distributions with measured radial microdrill data. A radial power depression factor, f, is determined using radial power distributions predicted by LASER. The factor f enters into the determination of the pellet centerline temperature, T, relative to the pellet surface temperature, T, through the expression:

$$\int_{T_0}^{T_c} K(T) dT = \frac{q'f}{4\pi}$$

where:

K(T) = the thermal conductivity for UO_2 with a uniform density distribution.

q' = the linear power generation rate.

4.4.2.11.3 Gap Conductance

The temperature drop across the pellet-clad gap is a function of the gap size and the thermal conductivity of the gas in the gap. The gap conductance model is selected such that when combined with the UO₂ thermal conductivity model, the calculated fuel centerline temperatures

reflect the in-pile temperature measurements. A more detailed discussion of the gap conductance model is presented in references 96 and 99.

4.4.2.11.4 Surface Heat Transfer Coefficients

The fuel rod surface heat transfer coefficients during subcooled forced convection and nucleate boiling are presented in paragraph 4.4.2.7.1.

4.4.2.11.5 Fuel Clad Temperatures

The outer surface of the fuel rod at the hotspot operates at a temperature of approximately 660°F for steady-state operation at rated power throughout core life due to the onset of nucleate boiling. Initially (beginning of life (BOL)), this temperature is that of the clad metal outer surface.

During operation over the life of the core, the buildup of oxides and crud on the fuel rod surface causes the clad surface temperature to increase. Allowance is made in the fuel center melt evaluation for this temperature rise. Since the thermal-hydraulic design basis limits DNB, adequate heat transfer is provided between the fuel clad and the reactor coolant so that the core thermal output is not limited by considerations of clad temperature.

4.4.2.11.6 Treatment of Peaking Factors

The total heat flux hot channel factor, FQ is defined by the ratio of the maximum to core average heat flux. The design value of FQ as discussed in paragraph 4.3.2.2.6 is 2.50 for normal operation. This results in a peak linear power of 14.47 kW/ft at full-power conditions.

As described in paragraph 4.3.2.2.6, the peak linear power resulting from overpower transients/operator errors [assuming a maximum overpower of 120 percent] does not exceed 22.4 kW/ft. The centerline fuel temperature must be below the UO₂ melt temperature over the lifetime of the rod, including allowances for uncertainties. The fuel temperature design basis is discussed in paragraph 4.4.1.2 and results in a maximum allowable calculated centerline temperature of 4700°F. The peak linear power for prevention of centerline melt is 22.4 kW/ft for VANTAGE 5 fuel and 22.5 kW/ft for LOPAR fuel. The centerline temperature at the peak linear power resulting from overpower transients/operator errors [assuming a maximum overpower of 120 percent] is below that required to produce melting.

4.4.3 DESCRIPTION OF THE THERMAL AND HYDRAULIC DESIGN OF THE RCS

4.4.3.1 Plant Configuration Data

Plant configuration data for the thermal-hydraulic and fluid systems external to the core are provided as appropriate in chapters 5, 6, and 9. Implementation of the ECCS is discussed in chapter 15. Some specific areas of interest are the following:

- A. Total coolant flowrates for the RCS and each loop are provided in table 5.1.2-1. Flowrates employed in the evaluation of the core are presented throughout section 4.4.
- B. Total RCS volume including pressurizer and surge line and RCS liquid volume including pressurizer water at steady-state power conditions are given in table 5.1.2-1.
- C. The flow path length through each volume may be calculated from physical data provided in the above-referenced table.
- D. The height of fluid in each component of the RCS may be determined from the physical data presented in section 5.4. The components of the RCS are water filled during power operation with the pressurizer being approximately 60-percent water filled.
- E. Components of the ECCS are to be located so as to meet the criteria for net positive suction head (NPSH) described in section 6.3.
- F. Line lengths and sizes for the safety injection system (SIS) are determined so as to guarantee a total system resistance which will provide, as a minimum, the fluid delivery rates assumed in the safety analyses described in chapter 15.
- G. The parameters for components of the RCS are presented in section 5.4.
- H. The steady-state pressure drops and temperature distributions through the RCS are presented in table 5.1.2-1.

4.4.3.2 Operating Restrictions on Pumps

The minimum NPSH and minimum seal injection flowrate must be established before operating the reactor coolant pumps. With the minimum 6-gal/min labyrinth seal injection flowrate established, the operator will have to verify that the system pressure satisfies NPSH requirements.

4.4.3.3 Power-Flow Operating Map (Boiling Water Reactor (BWR))

This paragraph is not applicable to VEGP.

4.4.3.4 <u>Temperature-Power Operating Map (PWR)</u>

The relationship between RCS temperature and power is shown in figure 4.4-8.

The effects of reduced core flow due to inoperative pumps is discussed in subsections 5.4.1 and 15.2.6 and section 15.3. Natural circulation capability of the system is discussed in paragraph 5.4.2.3.2.

4.4.3.5 <u>Load Following Characteristics</u>

Load follow using control rod motion and dilution or boration by the boron system is discussed in paragraph 4.3.2.4.16. The RCS is designed on the basis of steady-state operation at full-power heat load. The reactor coolant pumps utilize constant speed drives as described in section 5.4, and the reactor power is controlled to maintain average coolant temperature at a value which is a linear function of load, as described in section 7.7.

4.4.3.6 Thermal and Hydraulic Characteristics Summary Table

The thermal and hydraulic characteristics are given in tables 4.1-1, 4.4-1, and 4.4-2.

4.4.4 EVALUATION

4.4.4.1 Critical Heat Flux

The critical heat flux correlation utilized in the core thermal analysis is explained in detail in subsection 4.4.2.

4.4.4.2 Core Hydraulics

4.4.4.2.1 Flow Paths Considered in Core Pressure Drop and Thermal Design

The following flow paths for core bypass flow are considered:

- A. Flow through the spray nozzles into the upper head for head cooling purposes.
- B. Flow entering into the RCC guide thimbles.
- C. Leakage flow from the vessel inlet nozzle directly to the vessel outlet nozzle through the gap between the vessel and the barrel.
- D. Flow introduced between the baffle and the barrel for the purpose of cooling these components and not considered available for core cooling.
- E. Flow in the gaps between the fuel assemblies on the core periphery and the adjacent baffle wall.

The above contributions are evaluated to confirm that the design value of the core bypass flow is met. The design value of core bypass flow for the VEGP is equal to 6.4 percent of the total vessel flow.

Of the total allowance, 2.1 percent is associated with the core and the remainder is associated with the internals (items A, C, D, and E above). Calculations have been performed using

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drawing tolerances in the worst direction and accounting for uncertainties in pressure losses. Based on these calculations, the core bypass is no greater than the design value quoted above.

Flow model test results for the flow path through the reactor are discussed in paragraph 4.4.2.7.2.

4.4.4.2.2 Inlet Flow Distributions

Data has been considered from several one-seventh scale hydraulic reactor model tests⁽²²⁾⁽²³⁾⁽⁶¹⁾ in arriving at the core inlet flow maldistribution criteria to be used in the VIPRE-01 analyses (paragraph 4.4.4.5). THINC-I analyses made using this data have indicated that a conservative design basis is to consider a 5-percent reduction in the flow to the hot assembly.⁽⁶²⁾ The same design basis of 5-percent reduction to the hot assembly inlet is used in VIPRE-01 analyses.

The experimental error estimated in the inlet velocity distribution has been considered as outlined in reference 17, where the sensitivity of changes in inlet velocity distributions to hot channel thermal performance is shown to be small. Studies⁽¹⁷⁾ made with THINC-IV show that it is adequate to use the 5-percent reduction in inlet flow to the hot assembly for a loop out of service, based on the experimental data in references 22 and 23. VIPRE-01 was demonstrated to be equivalent to THINC-IV in reference 101.

The effect of the total flowrate on the inlet velocity distribution was studied in the experiments of reference 22. As was expected, on the basis of the theoretical analysis, no significant variation could be found in inlet velocity distribution with reduced flowrate.

4.4.4.2.3 Empirical Friction Factor Correlations

Empirical friction factor correlations are used in the VIPRE-01 code (described in paragraph 4.4.4.5).

The friction factor in the axial direction, parallel to the fuel rod axis, is evaluated using a correlation for the smooth tube ⁽¹⁰¹⁾. The effect of two-phase flow on the friction loss is expressed in terms of the single-phase friction pressure drop and a two-phase friction multiplier. The multiplier is calculated directly using the homogeneous equilibrium flow model.

The flow in the lateral directions, normal to the fuel rod axis, views the reactor core as a large tube bank. Thus, the lateral friction factor proposed by Idel'chik⁽²⁴⁾ is applicable. This correlation is of the form:

$$F_{L} = A \operatorname{Re}_{L}^{-0.2} \tag{11}$$

where:

A = a function of the rod pitch and diameter as given in reference 24.

Re_L = the lateral Reynolds number based on the rod diameter.

Extensive comparisons of VIPRE-01 predictions using these correlations to THINC-IV predictions are given in reference 101; they verify the applicability of these correlations in PWR design.

4.4.4.3 <u>Influence of Power Distribution</u>

The core power distribution, which is largely established at BOL by fuel enrichment, loading pattern, and core power level, is also a function of variables such as control rod worth and position, and fuel depletion through lifetime. Radial power distributions in various planes of the core are often illustrated for general interest; however, the core radial enthalpy rise distribution as determined by the integral of power up each channel is of greater importance for DNB analyses. These radial power distributions, characterized by $F_{\Delta H}^{N}$ (defined in paragraph 4.3.2.2.1), as well as axial heat flux profiles are discussed in the following two paragraphs.

4.4.4.3.1 Nuclear Enthalpy Rise Hot Channel Factor, $F_{\Lambda H}^{N}$

Given the local power density q' (kW/ft) at a point x, y, z in a core with N fuel rods and height H,

$$F_{\Delta H}^{N} = \frac{\text{hot rod power}}{\text{average rod power}} = \frac{\text{Max} \int_{O}^{H} q'(x_{o}, y_{o}, z_{o}) dz}{\frac{1}{N} \sum_{\substack{\text{all} \\ \text{rods}}} \int_{O}^{H} q'(x, y, z) dz}$$
(12)

The way in which $F_{\Delta H}^N$ is used in the DNB calculation is important. The location of minimum DNBR depends on the axial profile, and the value of DNBR depends on the enthalpy rise to that point. Basically, the maximum value of the rod integral is used to identify the most likely rod for minimum DNBR. An axial power profile is obtained which, when normalized to the design value of $F_{\Delta H}^N$, recreates the axial heat flux along the limiting rod. The surrounding rods are assumed to have the same axial profile with rod average powers which are typical distributions found in hot assemblies. In this manner, worst-case axial profiles can be combined with worst-case radial distributions for reference DNB calculations.

It should be noted again that $F_{\Delta H}^N$ is an integral and is used as such in DNB calculations. Local heat fluxes are obtained by using hot channel and adjacent channel explicit power shapes which take into account variations in horizontal power shapes throughout the core. The sensitivity of the DNB calculations to radial power shapes is discussed in reference 17. The VIPRE-01 analyses were based on the design radial power distributions discussed in reference 17.

For operation at a fraction of full power, the design $F_{\Delta H}^{N}$ used is given by:

$$F_{\Delta H}^{N} = F_{\Delta H}^{RTD} \left[1 + PF_{\Delta H} \left(1 - P \right) \right] \tag{13}$$

 $\mathsf{F}_{\Delta\mathsf{H}}^{\mathsf{RTP}}$ is the limit at rated thermal power (RTP) specified in the Core Operating Limits Report (COLR).

 $\mathsf{PF}_{\Delta H}$ is the power factor multiplier for $\mathsf{F}^{\mathsf{N}}_{\Delta H}$ specified in the COLR.

P is the fraction of rated thermal power.

The permitted relaxation of $F_{\Delta H}^{N}$ is included in the DNB protection setpoints and allows radial power shape changes with rod insertion to the insertion limits, ⁽⁶⁵⁾ thus allowing greater flexibility in the nuclear design.

4.4.4.3.2 Axial Heat Flux Distributions

As discussed in paragraph 4.3.2.2, the axial heat flux distribution can vary as a result of rod motion or power change or as a result of a spatial xenon transient which may occur in the axial direction. Consequently, it is necessary to measure the axial power imbalance by means of the excore nuclear detectors (as discussed in paragraph 4.3.2.2.7) and to protect the core from excessive axial power imbalance. The reference axial shape used in establishing core DNB limits (that is, overtemperature ΔT protection system setpoints) is a chopped cosine with a peak-to-average value of 1.55. The reactor trip system provides automatic reduction of the trip setpoints on excessive axial power imbalance. To determine the magnitude of the setpoint reduction, the reference shape is supplemented by other axial shapes skewed to the bottom and top of the core.

The course of those accidents in which DNB is a concern is analyzed in chapter 15 assuming that the protection setpoints have been set on the basis of these shapes. In many cases, the axial power distribution in the hot channel changes throughout the course of the accident due to rod motion, coolant temperature, and power level changes.

The initial conditions for the accidents for which DNB protection is required are assumed to be those permissible within the specified axial offset control limits described in paragraph 4.3.2.2. In the case of the loss-of-flow accident, the hot channel heat flux profile is very similar to the power density profile in normal operation preceding the accident. It is therefore possible to illustrate the calculated minimum DNBR for conditions representative of the loss-of-flow accident as a function of the flux difference initially in the core. A typical plot of this type is provided in figure 4.4-9. As noted on this figure, all power shapes were evaluated with a full-power radial peaking factor $F_{\Delta H}^{N}$ of 1.55. The radial contribution to the hot rod power shape is conservative both for the initial condition and for the condition at the time of minimum DNBR during the loss of flow transient. Also shown is the minimum DNBR calculated for the design power shape for nonoverpower/overtemperature DNB events. It can be seen that this design shape results in calculated DNBR that bounds all the normal operation shapes.

4.4.4.4 Core Thermal Response

A general summary of the steady-state thermal-hydraulic design parameters including thermal output, flowrates, etc., is provided in table 4.4-1.

As stated in subsection 4.4.1, the design bases of the application are to prevent DNB and to prevent fuel melting for Condition 1 and 2 events. The protective systems described in chapter 7 are designed to meet these bases. The response of the core to Condition 2 transients is given in chapter 15.

4.4.4.5 Analytical Methods

4.4.4.5.1 Core Analysis

The objective of reactor core thermal design is to determine the maximum heat-removal capability in all flow subchannels and to show that the core safety limits are not exceeded using the most conservative power distribution. The thermal design takes into account local variations in dimensions, power generation, flow redistribution, and mixing.

Prior to the power uprate to 3626 MWt, the THINC-IV code ⁽¹⁷⁾ (⁴⁸⁾ (⁶⁷⁾ (⁹⁸⁾ was used for the core thermal design. Commencing with the power uprate to 3626 MWt, the VIPRE-01 code is used for the core thermal design. VIPRE-01 is a three-dimensional subchannel code that has been developed to account for hydraulic and nuclear effects on the enthalpy rise in the core and hot channels ⁽¹⁰⁰⁾. VIPRE-01 modeling of a PWR core is based on one-pass modeling approach. ⁽¹⁰¹⁾ In the one-pass modeling, hot channels and their adjacent channels are modeled in detail, while the rest of the core is modeled simultaneously on a relatively coarse mesh. The behavior of the hot assembly is determined by superimposing the power distribution upon inlet flow distribution while allowing for flow mixing and flow distribution between flow channels. Local variations in fuel rod power, fuel rod and pellet fabrication, and turbulent mixing are also considered in determining conditions in the hot channels. Conservation equations of mass, axial and lateral momentum, and energy are solved for the fluid enthalpy, axial flow rate, lateral flow, and pressure drop.

4.4.4.5.2 Steady-State Analysis

The VIPRE-01 core model as approved by the NRC, reference 101, is used with the applicable DNB correlations to determine DNBR distributions along the hot channels of the reactor core under all expected operating conditions. The VIPRE-01 code is described in detail in reference 100, including discussions on code validation with experimental data. The VIPRE-01 modeling method is described in reference101, including empirical models and correlations used. The effect of crud on the flow and enthalpy distribution in the core is not directly accounted for in the VIPRE-01 evaluations. However, conservative treatment by the VIPRE-01 modeling method has been demonstrated to bound this effect in DNBR calculations (101).

The VIPRE-01 model has been demonstrated in reference 101 to be equivalent to the THINC-IV code. The DNBR limits for the DNB correlations in paragraph 4.4.2.2.1 remain unchanged with the VIPRE-01 model (101).

Estimates of uncertainties are discussed in paragraph 4.4.2.9.

4.4.4.5.3 Experimental Verification

Extensive additional experimental verification of VIPRE-01 is presented in reference 100.

The VIPRE-01 analysis is based on a knowledge and understanding of the heat transfer and hydrodynamic behavior of the coolant flow and the mechanical characteristics of the fuel

elements. The use of the VIPRE-01 analysis provides a realistic evaluation of the core performance and is used in the thermal analyses as described above.

4.4.4.5.4 Transient Analysis

VIPRE-01 is capable of transient DNB analysis. The conservation equations in the VIPRE-01 code contain the necessary accumulation terms for transient calculations. The input description can include one or more of the following time dependent arrays:

- 1. Inlet flow variation,
- Core heat flux variation.
- 3. Core pressure variation,
- 4. Inlet temperature or enthalpy variation.

At the beginning of the transient, the calculation procedure is carried out as in the steady-state analysis. The time is incremented by an amount determined either by the user or by the time step control options in the code itself. At each new time step the calculations are carried out with the addition of the accumulation terms which are evaluated using the information from the previous time step. This procedure is continued until a preset maximum time is reached.

At time intervals selected by the user, a complete description of the coolant parameter distributions as well as DNBR is printed out. In this manner the variation of any parameter with time can be readily determined.

The methods for evaluating fuel rod thermal response are described in subsection 15.0.11.

4.4.4.6 Hydrodynamic and Flow Power Coupled Instability

Boiling flow may be susceptible to thermo hydrodynamic instabilities. (68) These instabilities are undesirable in reactors, since they may cause a change in thermohydraulic conditions that may lead to a reduction in the DNB heat flux relative to that observed during a steady flow condition or to undesired forced vibrations of core components. Therefore, a thermohydraulic design criterion was developed which states that modes of operation under Condition 1 and 2 events shall not lead to thermo hydrodynamic instabilities.

Two specific types of flow instabilities are considered for VEGP operation. These are the Ledinegg (or flow excursion) type of static instability and the density wave type of dynamic instability.

A Ledinegg instability involves a sudden change in flowrate from one steady state to another. This instability occurs⁽⁶⁸⁾ when the slope of the RCS pressure drop-flowrate curve:

$$\left(\frac{\partial \Delta P}{\partial G}\right|_{GUM}$$

becomes algebraically smaller than the loop supply (pump head) pressure drop-flowrate curve:

$$\left(\frac{\partial \Delta P}{\partial G}\bigg|_{\text{external}}\right)$$

The criterion for stability is thus:

$$\left. \frac{\partial \Delta P}{\partial G} \right|_{\text{internal}} \ge \left. \frac{\partial \Delta P}{\partial G} \right|_{\text{external}}$$

The W pump head curve has a negative slope ($\partial \Delta P/\partial G$ external less than 0), whereas the RCS pressure drop-flow curve has a positive slope ($\partial \Delta P/\partial G$ internal greater than 0) over the Condition 1 and Condition 2 operational ranges. Thus, the Ledinegg instability will not occur.

The mechanism of density wave oscillations in a heated channel has been described by Lahey and Moody. Briefly, an inlet flow fluctuation produces an enthalpy perturbation. This perturbs the length and the pressure drop of the single-phase region and causes quality or void perturbations in the two-phase regions which travel up the channel with the flow. The quality and length perturbations in the two-phase region create two-phase pressure drop perturbations. However, since the total pressure drop across the core is maintained by the characteristics of the fluid system external to the core, then the two-phase pressure drop perturbation feeds back to the single-phase region. These resulting perturbations can be either attenuated or self-sustained.

A simple method has been developed by Ishii⁽⁷⁰⁾ for parallel closed-channel systems to evaluate whether a given condition is stable with respect to the density wave type of dynamic instability. This method had been used to assess the stability of typical Westinghouse reactor designs⁽⁷¹⁾⁽⁷²⁾⁽⁷³⁾ under Conditions 1 and 2 operation. The results indicate that a large margin to density wave instability exists; e.g., increases on the order of 150% of rated reactor power would be required for the predicted inception of this type of instability.

The application of the method of Ishii⁽⁷⁰⁾ to Westinghouse reactor designs is conservative due to the parallel open-channel feature of Westinghouse PWR cores. For such cores, there is little resistance to lateral flow leaving the flow channels of high-power density. There is also energy transfer from channels of high-power density to lower power density channels. This coupling with cooler channels has led to the opinion that an open-channel configuration is more stable than the above closed-channel analysis under the same boundary conditions. Flow stability tests⁽⁷⁴⁾ have been conducted where the closed channel systems were shown to be less stable than when the same channels were cross-connected at several locations. The cross-connections were such that the resistance to channel cross-flow and enthalpy perturbations would be greater than that which would exist in a PWR core which has a relatively low resistance to cross-flow.

Flow instabilities which have been observed have occurred almost exclusively in closed-channel systems operating at low pressures relative to the Westinghouse PWR operating pressures. Kao, Morgan, and Parker⁽⁷⁵⁾ analyzed parallel closed-channel stability experiments simulating a reactor core flow. These experiments were conducted at pressures up to 2200 psia. The results showed that for flow and power levels typical of power reactor conditions, no flow oscillations could be induced above 1200 psia.

Additional evidence that flow instabilities do not adversely affect thermal margin is provided by the data from the rod bundle DNB tests. Many Westinghouse rod bundles have been tested

over wide ranges of operating conditions with no evidence of premature DNB or of inconsistent data which might be indicative of flow instabilities in the rod bundle.

In summary, it is concluded that thermo hydrodynamic instabilities will not occur under Condition 1 and 2 modes of operation for Westinghouse PWR reactor designs. A large power margin exists to predicted inception of such instabilities. Analysis has been performed which shows that minor plant-to-plant differences in Westinghouse reactor designs such as fuel assembly arrays, core power-to-flow ratios, fuel assembly length, etc., will not result in gross deterioration of the above power margins.

4.4.4.7 <u>Fuel Rod Behavior Effects from Coolant Flow Blockage</u>

Coolant flow blockages can occur within the coolant channels of a fuel assembly or external to the reactor core. The effects of fuel assembly blockage within the assembly on fuel rod behavior are more pronounced than external blockages of the same magnitude. In both cases the flow blockages cause local reductions in coolant flow. The amount of local flow reduction, where it occurs in the reactor, and how far along the flow stream the reduction persists are considerations which will influence the fuel rod behavior. The effects of coolant flow blockages in terms of maintaining rated core performance are determined both by analytical and experimental methods. The experimental data are usually used to augment analytical tools such as computer programs similar to the THINC-IV or VIPRE-01 codes. Inspection of the DNB correlations (paragraph 4.4.2.2 and references 4, 84, 85, 86) shows that the predicted DNBR is dependent upon the local values of quality and mass velocity.

The VIPRE-01 code is capable of predicting the effects of local flow blockages on DNBR within the fuel assembly on a subchannel basis, regardless of where the flow blockage occurs. In reference 100, it is shown that for a fuel assembly similar to the Westinghouse design, VIPRE-01 accurately predicts the flow distribution within the fuel assembly when the inlet nozzle is completely blocked. Full recovery of the flow was found to occur about 30 in. downstream of the blockage. With the reactor operating at the nominal full-power conditions specified in table 4.4-1, the effects of an increase in enthalpy and decrease in mass velocity in the lower portion of the fuel assembly would not result in the reactor reaching the DNBR limit.

From a review of the open literature, it is concluded that flow blockage in open-lattice cores similar to the Westinghouse cores causes flow perturbations which are local to the blockage. For instance, Ohtsubo, et al., (76) show that the mean bundle velocity is approached asymptomatically about 4 in. downstream from a flow blockage in a single flow cell. Similar results were also found for two and three cells completely blocked. Basmer, et al., (78) tested an open-lattice fuel assembly in which 41 percent of the subchannels were completely blocked in the center of the test bundle between spacer grids. Their results show the stagnant zone behind the flow blockage essentially disappears after 1.65 L/De or about 5 in. for their test bundle. They also found that leakage flow through the blockage tended to shorten the stagnant zone or, in essence, the complete recovery length. Thus, local flow blockages within a fuel assembly have little effect on subchannel enthalpy rise. The reduction in local mass velocity is then the main parameter which affects the DNBR. If the plants were operating at full power and nominal steady-state conditions as specified in table 4.4-1, a reduction in local mass velocity greater than 60 percent in the VANTAGE + / VANTAGE 5 fuel would be required to reduce the DNBR to the DNBR limit. The above mass velocity effect on the DNB correlation was based on the assumption of fully developed flow along the full channel length. In reality a local flow blockage is expected to promote turbulence and thus would not likely affect DNBR at all.

Coolant flow blockages induce local cross-flows as well as promote turbulence. Fuel rod behavior is changed under the influence of a sufficiently high cross-flow component. Fuel rod vibration could occur, caused by this cross-flow component, through vortex shedding or turbulent mechanisms. If the cross- flow velocity exceeds the limit established for fluid elastic stability, large amplitude whirling results. The limits for a controlled vibration mechanism are established from studies of vortex shedding and turbulent pressure fluctuations. The cross-flow velocity required to exceed fluid elastic stability limits is dependent on the axial location of the blockage and the characterization of the cross-flow (jet flow or not). These limits are greater than those for vibratory fuel rod wear. Cross-flow velocity above the established limits can lead to mechanical wear of the fuel rods at the grid support locations. Fuel rod wear due to flow-induced vibration is considered in the fuel rod fretting evaluation (section 4.2).

4.4.5 TESTING AND VERIFICATION

4.4.5.1 Tests Prior to Initial Criticality

A reactor coolant flow test is performed following fuel loading but prior to initial criticality. Coolant loop elbow tap pressure data is obtained in this test. This data allows determination of the coolant flowrates at reactor operating conditions. This test verifies that proper coolant flowrates have been used in the core thermal and hydraulic analysis.

4.4.5.2 <u>Initial Power and Plant Operation</u>

Core power distribution measurements are made at several core power levels (chapter 14). These tests are used to ensure that conservative peaking factors are used in the core thermal and hydraulic analysis.

Additional demonstration of the overall conservatism of the THINC analysis was obtained by comparing THINC predictions to incore thermocouple measurements. These measurements were performed on the Zion reactor. No further inreactor testing is planned. In reference 101, the VIPRE-01 code was confirmed to be as conservative as the THINC code.

4.4.5.3 Component and Fuel Inspections

Inspections performed on the manufactured fuel are described in subsection 4.2.4. Fabrication measurements critical to thermal and hydraulic analysis are obtained to verify that the engineering hot channel factors in the design analyses (paragraph 4.4.2.2.4) are met.

4.4.6 INSTRUMENTATION REQUIREMENTS

4.4.6.1 <u>Incore Instrumentation</u>

The incore instrumentation system consists of chromel-alumel thermocouples at fixed core outlet positions and movable miniature neutron detectors (fission chambers) at selected fuel assemblies. The thermocouples are monitored by the plant safety monitoring system. The movable detectors can perform flux mapping at various core quadrants to obtain a flux map for any region of the core.

The thermocouples measure coolant outlet temperatures at preselected positions, and the fission chamber detectors positioned in guide thimbles which run the length of selected fuel assemblies measure the neutron flux distribution. Figure 4.4-10 shows the location of instrumented assemblies in the core. Provisions are made for 50 core exit thermocouples, divided into 2 trains of 25 each. Adequate provision has been made for inoperable or failed thermocouples because the number required for post accident monitoring functions described in subsections 7.5.3 and 7.5.4 are much less than 25 per channel. Information from the core exit thermocouples may be used to supplement information from the movable incore detectors, but it is not required for conducting flux maps and may not substitute for flux maps.

The incore instrumentation system is described in more detail in paragraph 7.7.1.9.

The movable incore detectors obtain data for the determination of incore fission power density distribution, coolant enthalpy distribution, and fuel burnup distribution. The core exit thermocouples provide input into the plant safety monitoring system (as described in appendix 4A) for indication of inadequate core cooling and core subcooling margin monitoring. The alternate shutdown panel also utilizes a core exit temperature signal from the core quadrants associated with loops 2 and 3.

The monitoring of core exit temperature following an accident is described in subsections 7.5.2 and 7.5.3. Core exit temperature is a category 1 parameter for monitoring types A, B, and C variables as indicated on table 7.5.4-1. This function is met by the two per quadrant per train requirement and reflected in the Technical Specifications. In accordance with NUREG-0737, the two thermocouples per train must be located such that they indicate the radial temperature gradient.

The use of core exit thermocouples as a backup to hot leg temperature measurement can be achieved by 16 thermocouples per train equally distributed across the core. Since no credit is taken for this function, table 7.5.2-1 indicates a need for eight thermocouples per train and two per quadrant.

4.4.6.2 Overtemperature and Overpower ΔT Instrumentation

The overtemperature ΔT trip protects the core against low DNBR. The overpower ΔT trip protects against excessive power (fuel rod rating protection).

As discussed in paragraph 7.2.1.1.2, factors included in establishing the overtemperature ΔT and overpower ΔT trip setpoints include the reactor coolant temperature in each loop and the axial distribution of core power through use of the two section excore neutron detectors.

4.4.6.3 Instrumentation to Limit Maximum Power Output

The output of the three ranges (source, intermediate, and power) of detectors, with the electronics of the nuclear instruments, is used to limit the maximum power output of the reactor within their respective ranges.

There are six radial locations containing a total of eight neutron flux detectors installed around the reactor in the primary shield. Fission chambers used for both source and intermediate range are installed on opposite "flat" portions of the core containing the startup sources at an elevation one-half of the core height. All four fission chambers are used for the source range. Two are used for intermediate range. Four dual-section uncompensated ionization chamber assemblies for the power range are installed vertically at the four corners of the core and located equidistant from the reactor vessel at all points and, to minimize neutron flux pattern distortions, within 1 ft of the reactor vessel. Each power range detector provides two signals corresponding to the neutron flux in the upper and in the lower sections of a core quadrant. The three ranges of detectors are used as inputs to monitor neutron flux from a completely shutdown condition to 120% of full power.

The output of the power range channels is used for:

- A. The rod speed control function.
- B. Alerting the operator to an excessive power unbalance between the quadrants.
- C. Protecting the core against the consequences of rod ejection accidents.
- D. Protecting the core against the consequences of adverse power distributions resulting from dropped rods.

Details of the neutron detectors and nuclear instrumentation design and the control and trip logic are given in chapter 7. The limits on neutron flux operation and trip setpoints are given in the Technical Specifications.

4.4.6.4 <u>Digital Metal Impact Monitoring System (DMIMS-DX™)</u>)

General System Description

The metal impact monitoring system ($DMIMS-DX^{TM}$) is designed to detect loose parts in the reactor coolant system. The system consists of sensors, preamplifiers, signal conditioners, signal processors, and a display. It contains 12 active instrument channels, each comprised of a piezoelectric accelerometer (sensor), signal conditioning and diagnostic equipment. Conformance with Regulatory Guide 1.133, Revision 1, is discussed in paragraph 1.9.133.2.

Two redundant sensors are fastened mechanically to the RCS at each of the following potential loose parts collection regions:

VEGP-FSAR-4

- Reactor pressure vessel: upper head region
- Reactor pressure vessel: lower head region
- Each steam generator: reactor coolant inlet region

The output signal from each accelerometer is passed through a preamplifier and an amplifier. The amplified signal is processed through a discriminator to eliminate noises and signals that are not indicative of loose parts. The processed signal is compared to a preset alarm setpoint. Loose parts detection is accomplished at a frequency of 1 kHz to 20 kHz, where background signals from the RCS are acceptable. Spurious alarming from control rod stepping is prevented by a module that detects CRDM motion commands and automatically inhibits alarms during control rod stepping.

If measured impact signals exceed the preset alarm level, audible and visible alarms in the control room are activated. Digital signal processors record the times that the first and subsequent impact signals reach various sensors. This timing information provides a basis for locating the loose part. The $DMIMS-DX^{TM}$ also has a provision for audio monitoring of any channel. The audio signal can be compared to a previously recorded audio signal, if desired.

The online sensitivity of the $DMIMS-DX^{TM}$ is such that the system will detect a loose part that weighs from 0.25 to 30 lb and impacts with a kinetic energy of 0.5 ft-lb on the inside surface of the RCS pressure boundary within 3 ft of a sensor.

The *DMIMS-DX*TM audio and visual alarm capability will remain functional after an Operating Basis Earthquake (OBE). All of the *DMIMS-DX*TM components are qualified for structural integrity during a Safe Shutdown Earthquake (SSE) and will not mechanically impact any safety-related equipment. In addition, the equipment inside containment is designed to remain functional through normal radiation exposures anticipated during a 40-year operating lifetime^a. Physical separation of the two instrument channels, associated with the redundant sensors at each reactor coolant system location, exists from each sensor to the output of the incontainment signal conditioning devices. The incontainment signal conditioning devices are accessible during power operation. The *DMIMS-DX*TM components outside containment are located in a mild environment. Capabilities exist for subsequent periodic online channel checks and channel functional tests and for offline channel calibrations at refueling outages. Figure 4.4-11 shows a block diagram of the loose parts monitoring system.

Key Features, Components and Architecture

Key features of system components and architecture are discussed in the following sections.

Sensors

The sensors are piezoelectric accelerometers that convert acceleration to electric charge. The acoustic waves created by an impacting metallic object can be detected by the piezoelectric accelerometers. While the excitation of the impact produces a very wideband frequency response, the frequency range of interest for most loose parts is 1 kHz to 20 kHz.

^a The operating licenses for both VEGP units have been renewed and the original licensed operating terms have been extended by 20 years. In accordance with 10 CFR Part 54, appropriate aging management programs and activities have been initiated to manage the detrimental effects of aging to maintain functionality during the period of extended operation (see chapter 19).

Piezoelectric accelerometers are high output impedance devices that convert acceleration to electric charge. The flat frequency response range for the accelerometers used in DMIMS- DX^{TM} is from 5 Hz to 10 kHz, and they have a useful frequency upper limit of over 20 kHz. The resonant frequency of the accelerometers is greater than 30 kHz. The accelerometers are designed to operate at high temperature (nominally 625 °F) and have high radiation capability.

The piezoelectric elements in the accelerometers are electrically isolated from the component to which they are attached in order to prevent unwanted noise due to ground loops. The accelerometers typically have an integral 4 foot mineral-insulated ("hardline") cable and a large triax connector. This hardline cable is also built to withstand high temperatures, while the connector allows for interfacing to lower temperature softline cables.

Softline Cable

Because the charge output of an accelerometer is a very low level signal, and normal cables can emit charge upon being vibrated, a special low-noise, radiation-resistant softline cable is used between the accelerometer and preamplifier.

Preamplifier

The remote preamplifier is mounted in a sealed metal enclosure inside containment. The charge signal from the accelerometer is converted to a voltage signal. The preamplifier operates in a "charge" amplifier mode such that the capacitance of the cable between the high-output-impedance accelerometer and the preamplifier has very little effect on the signal or its calibration. The charge preamplifier output voltage is then a normal, low-impedance millivolt instrument signal requiring only normal cabling and shielding considerations.

Signal Conditioner

The signal conditioner module provides power to the remote preamplifier, provides final amplification of the signal to a calibrated full scale range, and provides lowpass and highpass filtering.

Audio Subsystem

The audio patch panel, audio amplifier, and speakers make up the audio subsystem. Listening by a trained ear can be a very effective tool for evaluation and validation of signal characteristics. The system is designed such that any channel may be selected at any time for audio monitoring.

Digital Signal Processing (DSP) Processor

In the Digital Signal Processing (DSP) processor, the signals are converted from analog to digital at a high rate, and the impact detection algorithm is applied by a special microprocessor optimized for digital signal processing. The board contains a buffer memory that can store the complete impact signal time history for its monitored channels. Upon the detection of an impact, the data are normally transferred to the main Central Processing Unit (CPU) process for further evaluation, waveform storage, and alarm generation. However, if for some reason the CPU processor fails, the DSP processor has the capability for generating alarms on its own.

Central Processing Unit (CPU) Processor

The CPU processor is a personal computer architecture device. It takes the data from the DSP processors, controls the mass storage devices, provides displays of monitoring system information, drives the printer, and generates alarms. The CPU uses a PCI bus for high speed communication with the other processor modules and drives the tape and disk peripherals by means of a parallel Small Computers System Interface (SCSI) interface. Addition of the peripherals provides for mass data storage onto high speed digital tape and writeable CDs.

Display

The display is a qualified, high-resolution, color panel that is overlaid with a high-resolution touchscreen surface. The display shows the system and alarm statuses at a glance, presents the waveforms used in impact analysis, and shows the analysis conclusions. By means of the touchscreen, which has all of the capabilities of a standard mouse, many system functions can be run without opening the keyboard drawer.

Alarm Panel

The alarm panel provides continuous indication of alarm or trouble status, allowing the color display to be turned off when not being viewed. The panel contains red LEDs for alarm indication, orange LEDs for trouble indication, yellow LEDs that flash each time an impact event is detected by their respective channels, and green LEDs for indication of proper DSP operation.

Printer

A high-resolution laser printer is provided for printout of system status, waveform graphs, and other data for the generation of reports.

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TABLE 4.4-1 (SHEET 1 OF 3)

THERMAL AND HYDRAULIC COMPARISON TABLE

Design	n Parameters	LOPAR ^(m)	VANTAGE + / VANTAGE 5
Reactor core heat output (MWt) Reactor core heat output (10 ⁶ Btu/h)		3565 ^(m) 12,164	3626 ^(l) 12,372
Heat generated in fuel (%) System pressure, nominal (psia) System pressure, minimum steady-state (psia)		97.4 2250 2200	97.4 2250 2200
Minimu	um DNBR at nominal conditions Typical flow channel	3.20	2.45
	Thimble (cold wall) flow channel	3.01	2.35
Minimu	um DNBR for design transients Typical flow channel Thimble (cold wall) flow channel	1.23 1.22	1.24 1.23
DNB correlation ^(a)		WRB-1	WRB-2
Coolar	nt conditions ^(b) Vessel minimum measured flowrate(MMF) ^(c) 10 ⁶ lbm/h gpm	142.9 384,000	143.0 384,000
Vessel lbm/h	thermal design flowrate (TDF) 10 ⁶	139.4 374,400	139.5 374,400
Effectiv	ve flowrate for heat transfer (based	130.5	130.6
	on TDF) 10 ⁶ lbm/h gpm	350,440	350,440
Effective flow area for heat transfer (ft²)(d)		51.08	54.13
Average velocity along fuel rods (ft/s) ^(d)		16.3	15.3
Averaç	ge mass velocity (10 ⁶ lbm/h-ft²) (based on TDF) ^(d)	2.55	2.41

TABLE 4.4-1 (SHEET 2 OF 3)

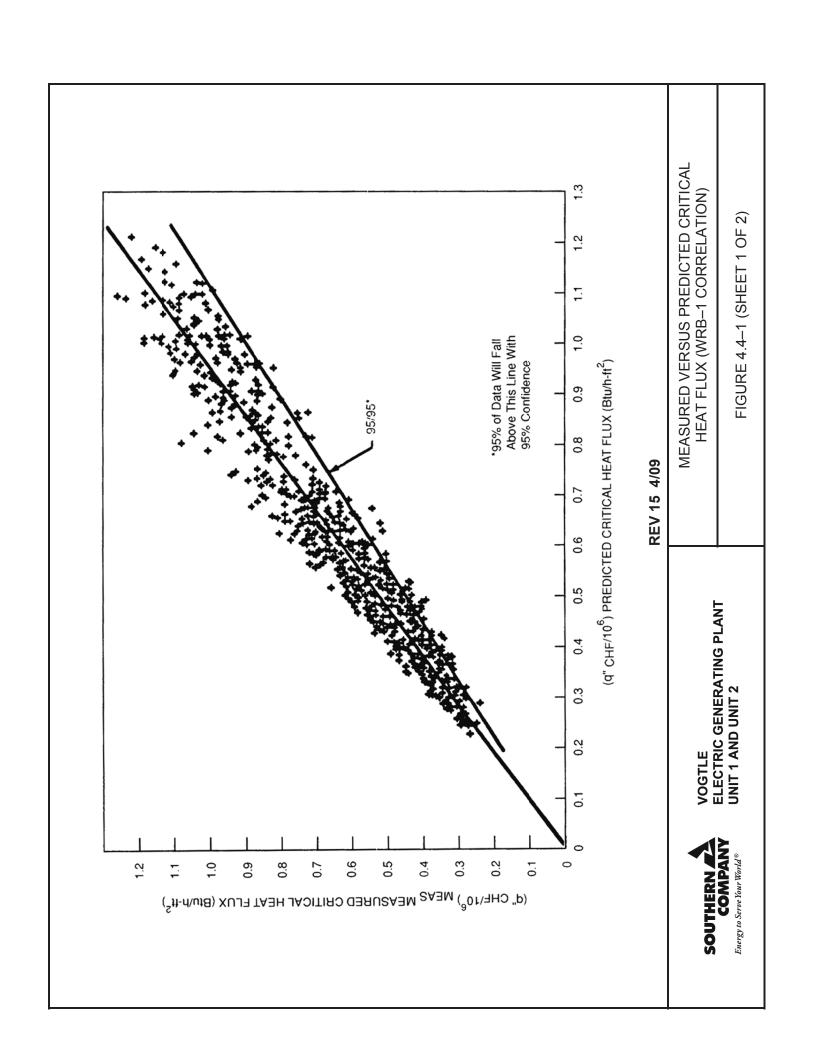
<u>Design Parameters</u>	LOPAR ^(m)	VANTAGE + / VANTAGE 5
Coolant temperature Nominal inlet (°F) Average rise in vessel (°F)	556.8 63.2	556.3 64.2
Average rise in core (°F)	66.9	68.0
Average in core (°F)	592.2	592.3
Average in vessel (°F)	588.4	588.4
Heat transfer Active heat transfer surface area (ft²)(d)	59,742	57,505
Average heat flux (Btu/h-ft²)(d)	198,370	209,612
Maximum heat flux for normal $^{(d,e)}$ operation (Btu/h-ft 2)	495,925	524,030
Average linear power (kW/ft) ^(f)	5.69	5.788
Peak linear power for normal operation (kW/ft) ^(e,f)	14.2	14.47
Peak linear power resulting from overpower transients/operator errors, assuming a maximum overpower of 120%(kW/ft)	<22.4	<22.4
Peak linear power for prevention of centerline melt (kW/ft) ^(h)	22.5	22.4
Power density (kW/l of core)(i)	109.2	111.1
Specific power (kW/kg uranium) ^(d,i)	39.0	43.2
Fuel central temperature Peak at peak linear power for prevention of centerline melt (°F)	4,700	4,700
Pressure drop Across core (psi) ^(d) Across vessel, including nozzle (psi) ^(d)	23.3 ± 2.3 46.2 ± 4.6	$28.6 \pm 2.9^{(j,k)}$ $48.5 \pm 4.9^{(k)}$

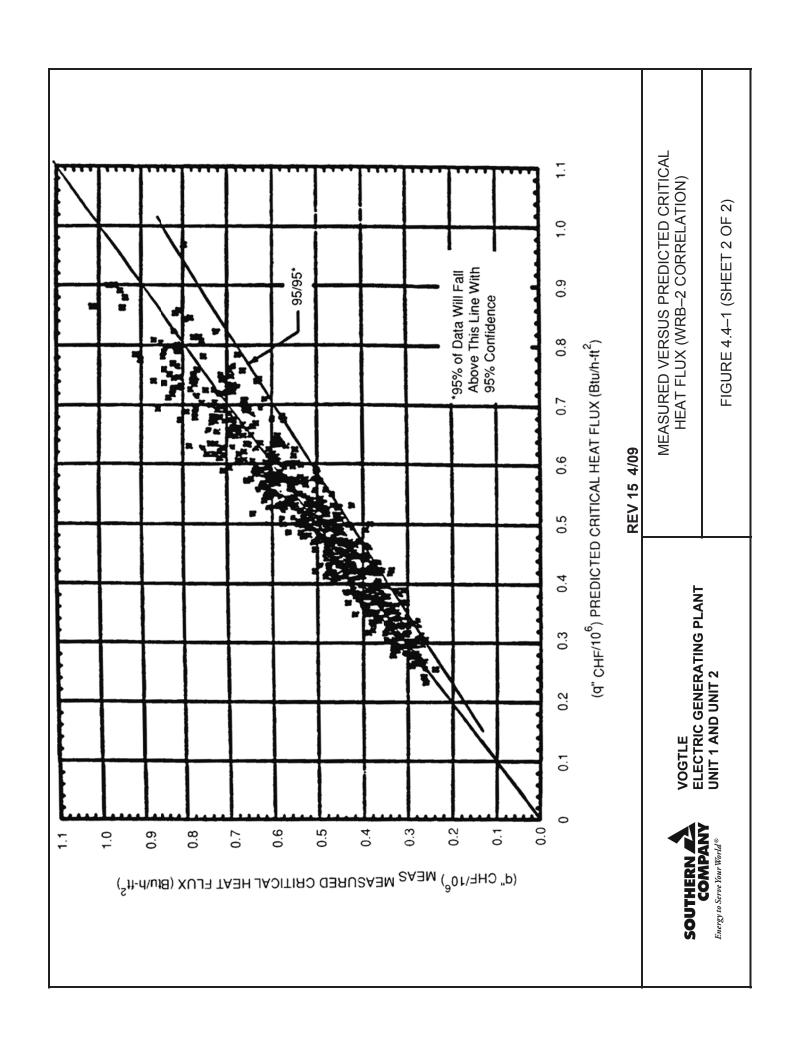
TABLE 4.4-1 (SHEET 3 OF 3)

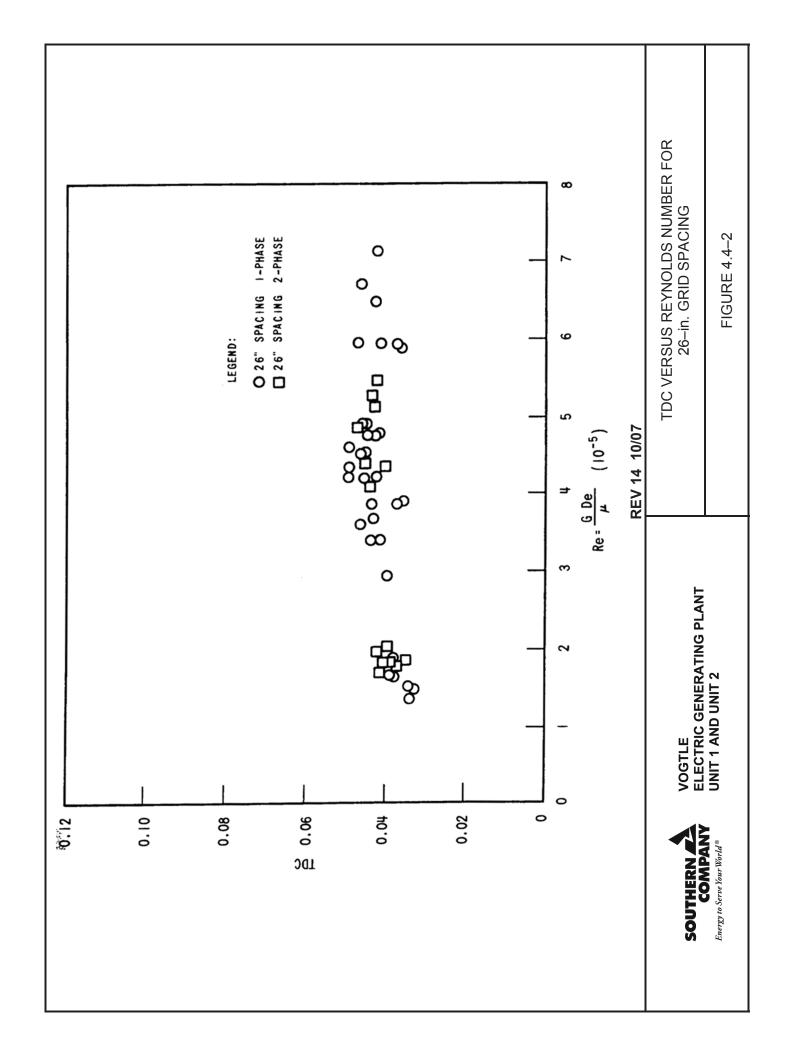
- a. See paragraph 4.4.2.2.1 for the use of the W-3 correlation.
- b. Flowrates are based on 10 percent steam generator tube plugging.
- c. Inlet temperature = 557.0°F
- d. Assumes all LOPAR or VANTAGE + / VANTAGE 5 Core.
- e. Based on 2.50 F_Q peaking factor.
- f. Based on densified active fuel length.
- g. See paragraph 4.3.2.2.6
- h. See paragraph 4.4.2.11.6.
- i. Based on cold dimensions and 95 percent of theoretical density fuel.
- j. Based on a best-estimate reactor flowrate of 102,100 gpm/loop and an inlet temperature of 558.8°F. The pressure drop for the PRIME™ fuel assembly design is bounded by the AOR pressure drop associated with the best-estimate flowrate noted above.
- k. With thimble plugs.
- I. The VEGP MUR 1.7% power uprate increases the licensed reactor core power level from 3565 MWt to 3625.6 MWt.
- m. LOPAR fuel is not analyzed for the power uprate to 3626 MWt. The LOPAR values at 3565 MWt are retained for historical purposes.

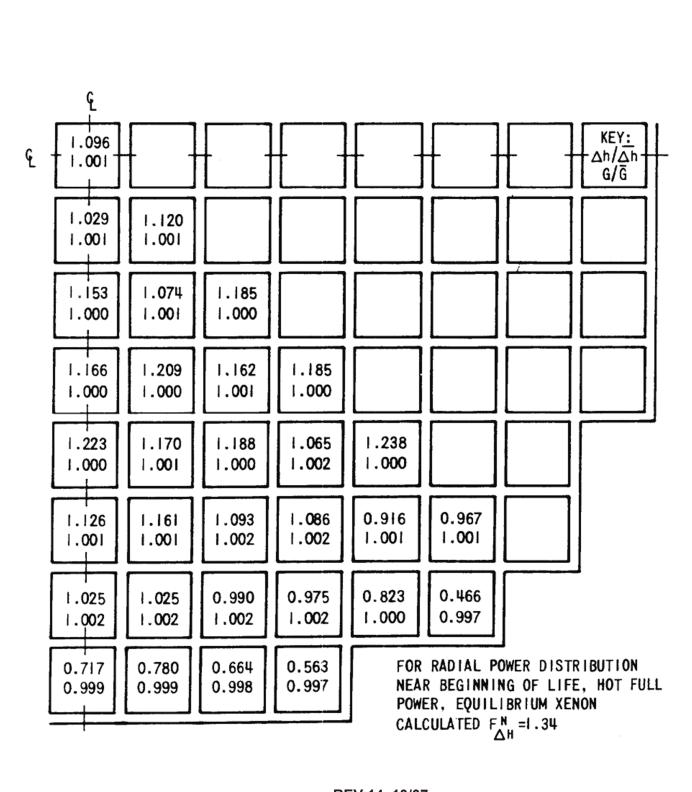
TABLE 4.4-2
VOID FRACTIONS AT NOMINAL REACTOR CONDITIONS

		<u>Average</u>	<u>Maximum</u>	
Core	(VANTAGE + / VANTAGE 5)	< 0.01%	- -	
Hot subchannel	(VANTAGE + / VANTAGE 5)	1.8%	7.0%	







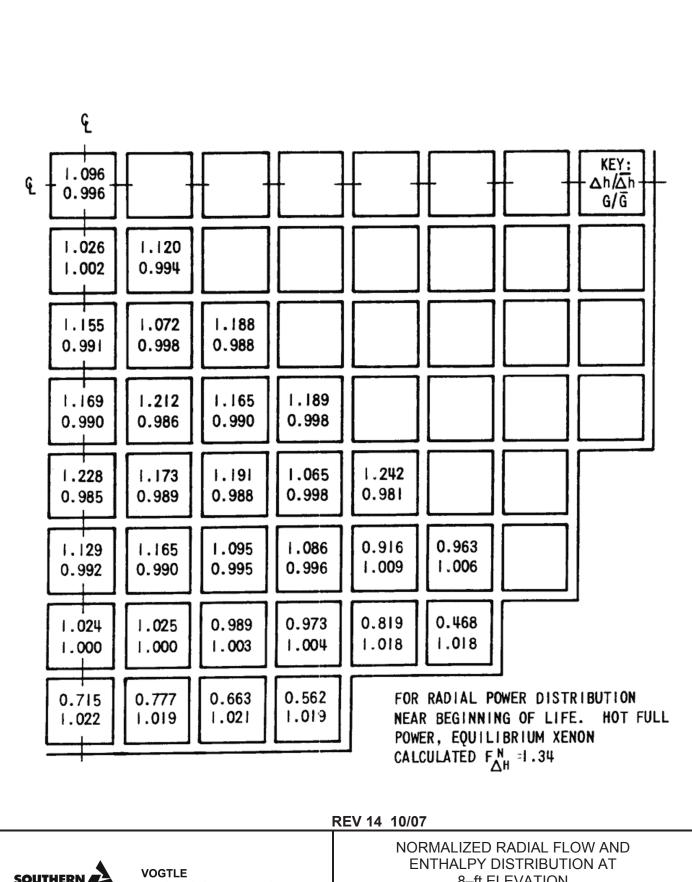


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VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 NORMALIZED RADIAL FLOW AND ENTHALPY DISTRIBUTION AT 4-ft ELEVATION

FIGURE 4.4–3

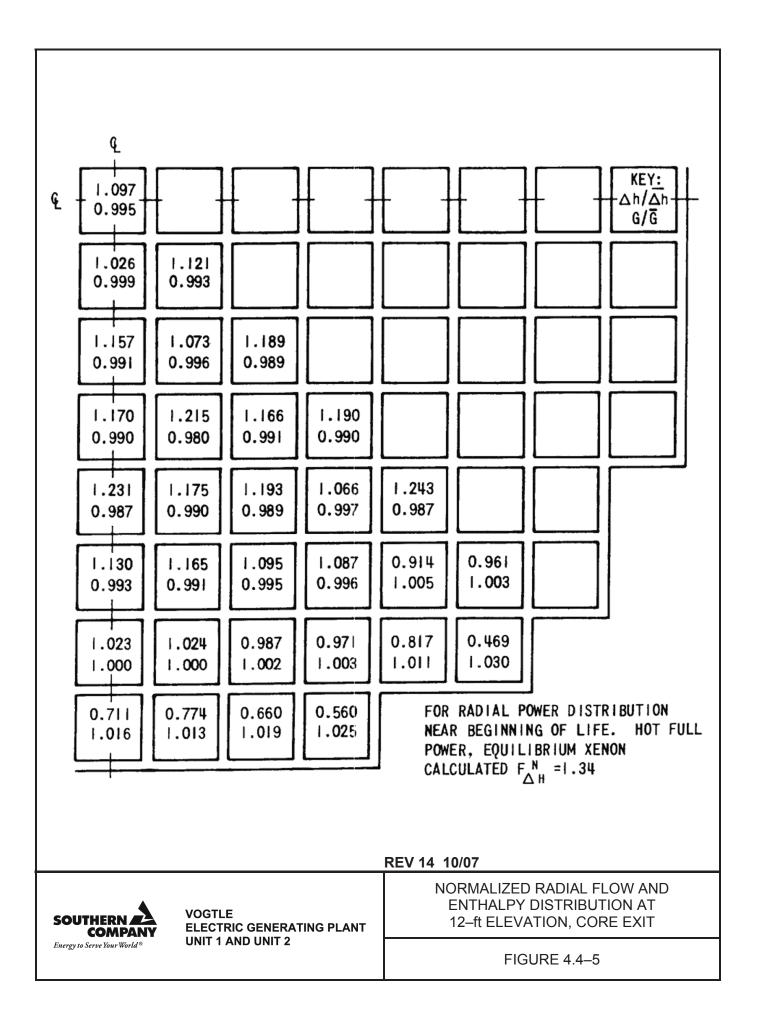


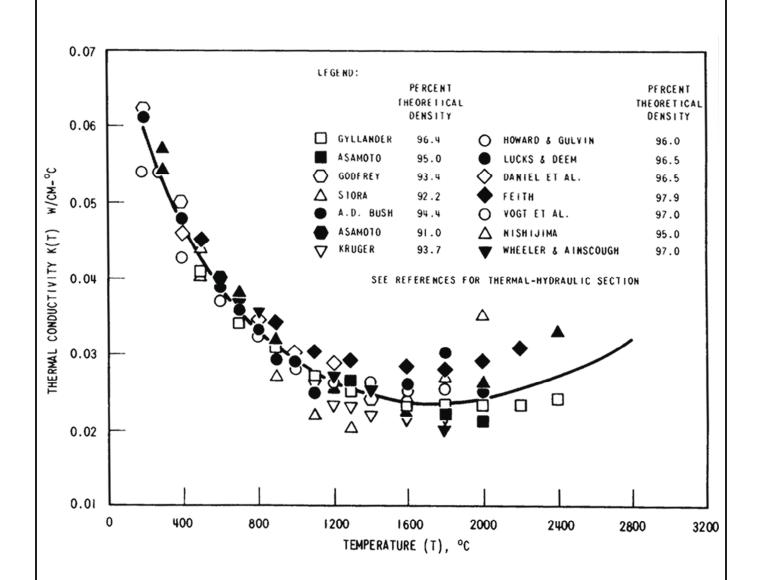


ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2

8-ft ELEVATION

FIGURE 4.4–4





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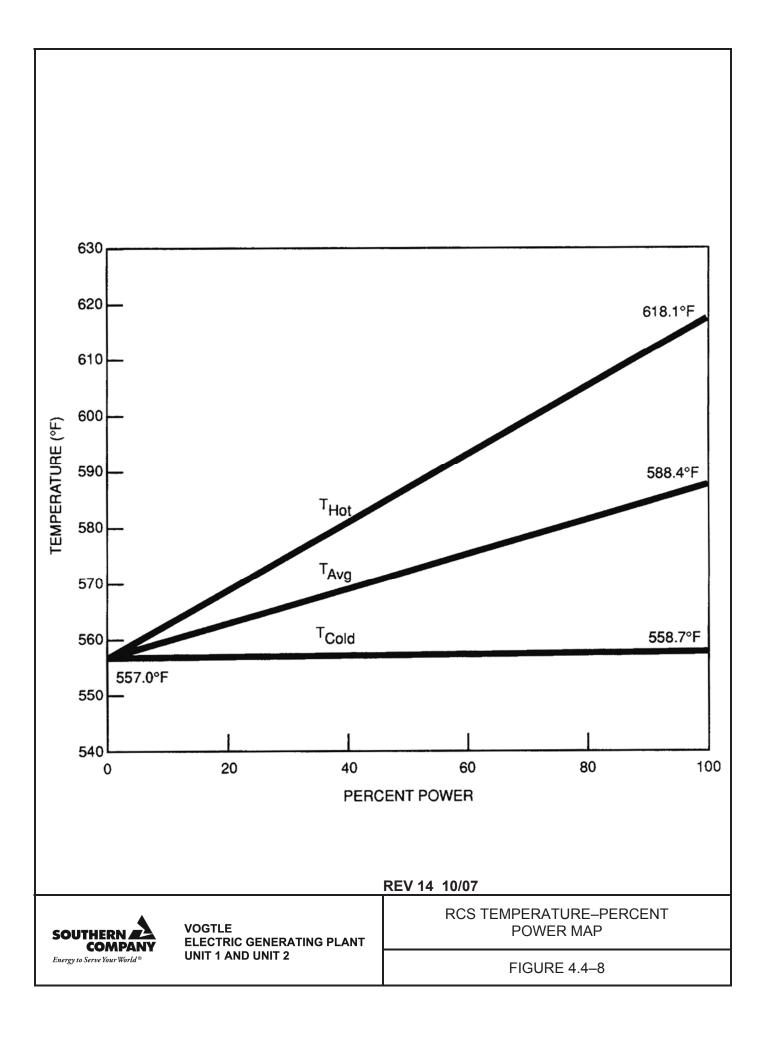


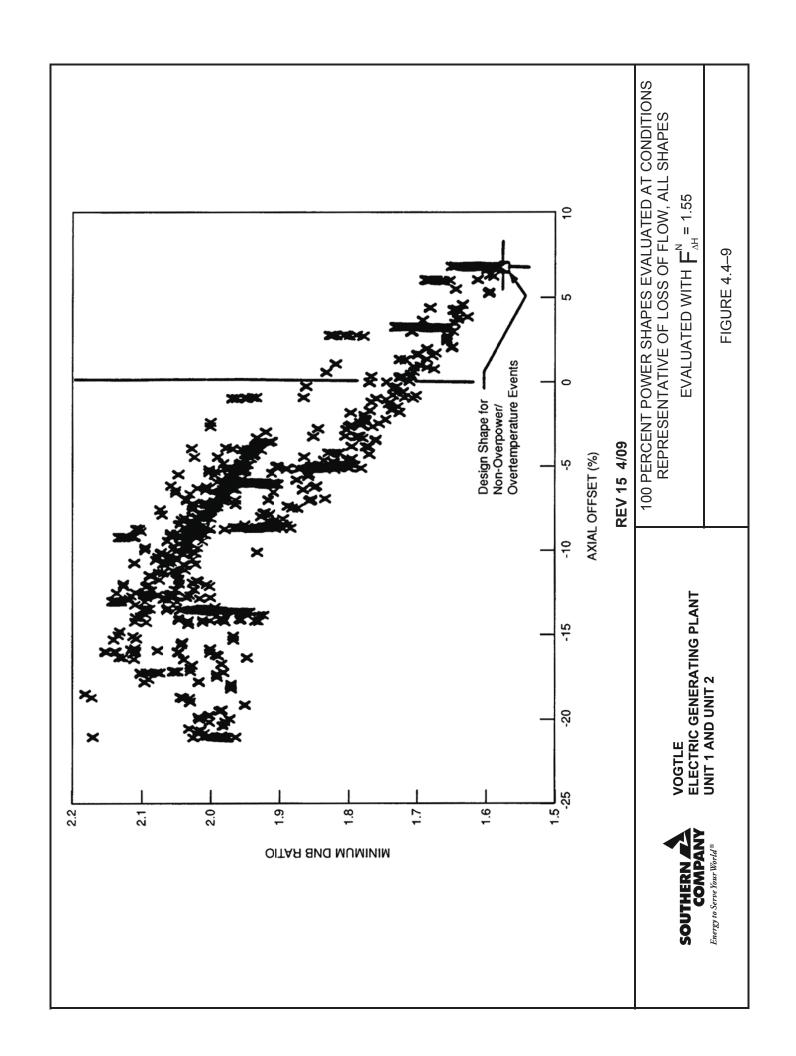
VOGTLE
ELECTRIC GENERATING PLANT
UNIT 1 AND UNIT 2

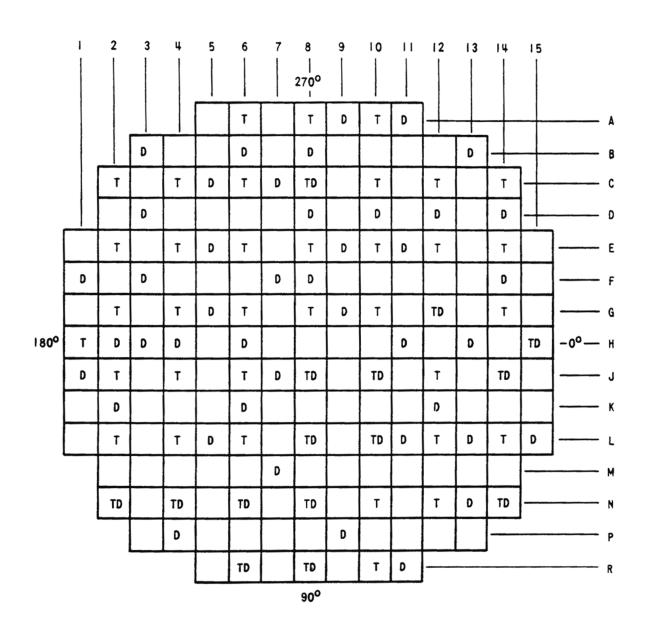
THERMAL CONDUCTIVITY OF UO₂ (DATA CORRECTED TO 95% THEORETICAL DENSITY)

FIGURE 4.4-6

FIGURE 4.4–7	UNIT 1 AND UNIT 2	Energy to Serve Your World ®
DELETED	VOGTLE ELECTRIC GENERATING PLANT	SOUTHERN
REV 15 4/09		
TED	DELETED	







T = THERMOCOUPLE LOCATIONS

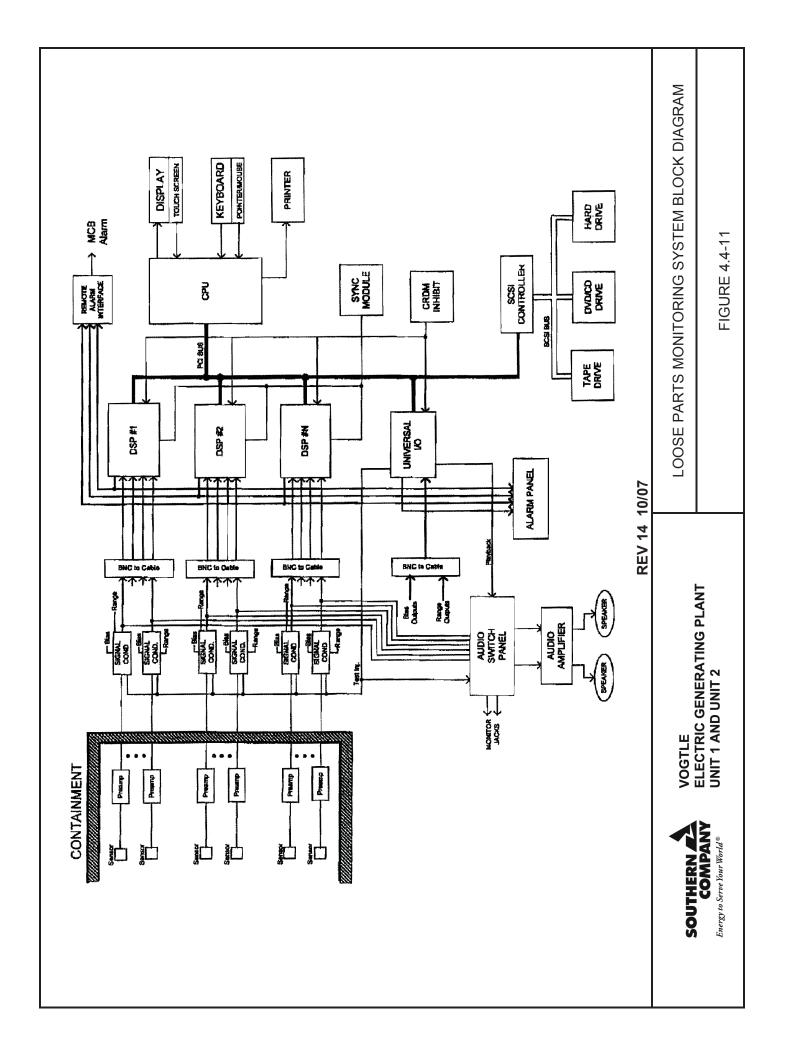
D = MOVABLE INCORE DETECTOR LOCATIONS

REV 14 10/07



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 DISTRIBUTION OF INCORE INSTRUMENTATION

FIGURE 4.4-10



4.5 REACTOR MATERIALS

4.5.1 CONTROL ROD DRIVE SYSTEM STRUCTURAL MATERIALS

4.5.1.1 Materials Specifications

All parts exposed to reactor coolant are made of metals which resist the corrosive action of the water. Three types of metals are used exclusively: stainless steels, nickel-chromium-iron, and cobalt-based alloys. In the case of stainless steels, only austenitic and martensitic stainless steels are used. For pressure boundary parts, martensitic stainless steels are not used in the heat-treated conditions which cause susceptibility to stress-corrosion cracking or accelerated corrosion in Westinghouse pressurized water reactor chemistry. Pressure boundary parts/components are made of type 304 or equivalent material.

Internal latch assembly parts are fabricated of heat-treated martensitic stainless steel. Heat treatment is such that susceptibility to stress-corrosion cracking is not initiated.

A. Pressure Vessel

All pressure-containing materials comply with Section III of the American Society of Mechanical Engineers (ASME) Boiler and Pressure Vessel (B&PV) Code and are fabricated from austenitic (type 304) stainless steel.

B. Coil Stack Assembly

The coil housings require a magnetic material. Both low carbon cast steel and ductile iron have been successfully tested for this application. The choice, made on the basis of cost, indicates that ductile iron will be specified on the control rod drive mechanism (CRDM). The finished housings are zinc plated or flame sprayed to provide corrosion resistance.

Coils are wound on bobbins of molded Dow Corning type 302 material, with double glass insulated copper wire. Coils are then vacuum impregnated with silicon varnish. A wrapping of mica sheet is secured to the coil outside diameter. The result is a well-insulated coil capable of sustained operation at 200°C.

C. Latch Assembly

Magnetic pole pieces are fabricated from type 410 stainless steel. All nonmagnetic parts, except pins and springs, are fabricated from type 304 stainless steel. Haynes-25 is used to fabricate link pins. Springs are made from nickel-chromium-iron alloy (Inconel-X). Latch arm tips are clad with Stellite-6 to provide improved wearability. Hard chrome plate and Stellite-6 are used selectively for bearing and wear surfaces.

D. Drive Rod Assembly

The drive rod assembly utilizes a type 410 stainless steel drive rod. The coupling is machined from type 403 stainless steel. Other parts are type 304 stainless steel with the exception of the springs, which are nickel-chromium-iron alloy; the locking button, which is Haynes-25; and the belleville washers, which are Inconel-718. Several small parts (screws and pins) are Inconel-600.

4.5.1.2 <u>Fabrication and Processing of Austenitic Stainless Steel Components</u>

The discussions provided in subsection 5.2.3, concerning the processes, inspections, and tests on austenitic stainless steel components to ensure freedom from increased susceptibility to intergranular corrosion caused by sensitization, and the discussions provided in subsection 5.2.3, concerning the control of welding of austenitic stainless steels especially control of delta ferrite, are applicable to the austenitic stainless steel pressure-housing components of the CRDM.

4.5.1.3 <u>Contamination Protection and Cleaning of Austenitic Stainless Steel</u>

The CRDMs are cleaned prior to delivery in accordance with the guidance of American National Standards Institute (ANSI) 45.2.1. Process specifications in packaging and shipment are discussed in subsection 5.2.3. Westinghouse personnel conduct surveillance of these operations to ensure that manufacturers and installers adhere to appropriate requirements as discussed in subsection 5.2.3.

4.5.1.4 Other Materials

Haynes-25 is used in small quantities to fabricate link pins. The material is ordered in the solution-treated, cold-worked condition. Stress-corrosion cracking has not been observed in this application over the last 15 years.

The CRDM springs are made from nickel-chromium-iron alloy (Inconel-750) ordered to MIL-S-23192 or MIL-N-24114 Class A No. 1 temper-drawn wire. Operating experience has shown that springs made of this material are not subject to stress-corrosion cracking.

4.5.2 REACTOR INTERNALS MATERIALS

4.5.2.1 Materials Specifications

All the major material for the reactor internals is type 304 stainless steel. Parts not fabricated from type 304 stainless steel include bolts and dowel pins, which were fabricated from type 316 stainless steel, and radial support key bolts, which were fabricated of Inconel-750. Radial support clevis inserts are Inconel-600, and the holddown spring is type 403 stainless steel. These materials are listed in table 5.2.3-2. There are no other materials used in the reactor internals or core support structures which are not otherwise included in the ASME Code, Section III, Appendix I.

4.5.2.2 Controls on Welding

The discussions provided in subsection 5.2.3 are applicable to the welding of reactor internals and core support components.

4.5.2.3 <u>Nondestructive Examination of Tubular Products and Fittings</u>

The nondestructive examination of wrought seamless tubular products and fittings is in accordance with Section III of the ASME Code.

4.5.2.4 Fabrication and Processing of Austenitic Stainless Steel Components

The discussions provided in subsection 5.2.3 and section 1.9 verify conformance of reactor internals and core support structures with Regulatory Guide 1.44.

The discussions provided in subsection 5.2.3 and section 1.9 verify conformance of reactor internals and core support structures with Regulatory Guide 1.31.

The discussion provided in section 1.9 verifies conformance of reactor internals with Regulatory Guide 1.34.

The discussion provided in section 1.9 verifies conformance of reactor internals and core support structures with Regulatory Guide 1.71.

4.5.2.5 Contamination Protection and Cleaning of Austenitic Stainless Steel

The discussions provided in subsection 5.2.3 and section 1.9 are applicable to the reactor internals and core support structures and verify conformance with ANSI 45 specifications and Regulatory Guide 1.37.

4.6 FUNCTIONAL DESIGN OF REACTIVITY CONTROL SYSTEMS

4.6.1 INFORMATION FOR CONTROL ROD DRIVE SYSTEM

The control rod drive system (CRDS) is described in paragraph 3.9.4.1. Figures 3.9.4-1 and 3.9.4-2 provide the details of the control rod drive mechanisms, and figure 4.2-8 provides the layout of the CRDS. No hydraulic system is associated with its functioning. The instrumentation and controls for the reactor trip system are described in section 7.2, and the reactor control system is described in section 7.7.

4.6.2 EVALUATIONS OF THE CRDS

The CRDS has been analyzed in detail in the failure mode and effects analysis.⁽¹⁾ This study and the analyses presented in chapter 15 demonstrate that the CRDS performs its intended safety function, a reactor trip, by putting the reactor in a subcritical condition when a safety system setting is reached, with any assumed credible failure of a single active component. The essential elements of the CRDS (those required to ensure reactor trip) are isolated from nonessential portions of the CRDS (the rod control system) as described in section 7.2. The essential portion of the CRDS is protected from the effects of postulated moderate- and high-energy line breaks.

Despite the extremely low probability of a common mode failure impairing the ability of the reactor trip system to perform its safety function, analyses have been performed in accordance with the requirements of WASH-1270. These analyses, documented in references 2 and 3, have demonstrated that acceptable safety criteria would not be exceeded even if the CRDS were rendered incapable of functioning during a reactor transient for which its function would normally be expected.

The design of the control rod drive mechanism (CRDM) is such that failure of the CRDM cooling system will, in the worst case, result in an individual control rod trip or a full reactor trip (section 7.2).

4.6.3 TESTING AND VERIFICATION OF THE CRDS

The CRDS is extensively tested prior to its operation. These tests may be subdivided into five categories:

- Prototype tests of components.
- Prototype CRDS tests.
- Production tests of components following manufacture and prior to installation.
- Onsite preoperational and initial startup tests.
- Periodic inservice tests.

These tests, which are described in paragraph 3.9.4.4, sections 4.2 and 14.2, and chapter 16, are conducted to verify the operability of the CRDS when called upon to function.

4.6.4 INFORMATION FOR COMBINED PERFORMANCE OF REACTIVITY SYSTEMS

As is indicated in chapter 15, the only postulated events which assume credit for reactivity control systems, other than a reactor trip to render the plant subcritical, are the steam line break, feedwater line break, and loss-of-coolant accident (LOCA). The reactivity control systems for which credit is taken in these accidents are the reactor trip system and the safety injection system (SIS). Additional information on the CRDS is presented in subsection 3.9.4 and on the SIS in section 6.3. Note that no credit is taken for the boration capabilities of the chemical and volume control system (CVCS) as a system in the analysis of transients presented in chapter 15. Information on the capabilities of the CVCS is provided in subsection 9.3.4. The adverse boron dilution possibilities due to the operation of the CVCS are investigated in subsection 15.4.6. Prior proper operation of the CVCS has been presumed as an initial condition to evaluate transients, and appropriate technical specifications and requirements in the Technical Requirements Manual have been prepared to ensure the correct operation or remedial action.

4.6.5 EVALUATION OF COMBINED PERFORMANCE

The evaluation of the steam line break, the feedwater line break, and the LOCA, which presumes the combined actuation of the reactor trip system to the CRDS and the SIS, is presented in subsections 15.1.5, 15.2.8, and 15.6.5. Reactor trip signals and safety injection signals for these events are generated from functionally diverse sensors and actuate diverse means of reactivity control, i.e., control rod insertion and injection of soluble poison.

Nondiverse but redundant types of equipment are utilized only in the processing of the incoming sensor signals into appropriate logic which initiates the protective action. This equipment is described in detail in sections 7.2 and 7.3. In particular, note that protection from equipment failures is provided by redundant equipment and periodic testing. Effects of failures of this equipment have been extensively investigated. The failure mode and effects analysis described in reference 4 verifies that any single failure will not have a deleterious effect upon the engineered safety features actuation system. Adequacy of the emergency core cooling system and SIS performance under faulted conditions is verified in section 6.3.

4.6.6 REFERENCES

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APPENDIX 4A

RESPONSE TO NUREG-0737, II.F.2, INSTRUMENTATION FOR DETECTION OF INADEQUATE CORE COOLING

4A.1 THE INADEQUATE CORE COOLING (ICC) MONITORING SYSTEM INSTALLED AT VEGP WILL INCLUDE THE FOLLOWING:

- Core exit thermocouple (T/C) monitoring.
- Core subcooling margin monitoring.
- Reactor vessel level monitoring.

A detailed electrical and layout description of each of the above ICC monitoring subsystems is given below.

A. Core Exit Thermocouple System

The core exit thermocouple monitoring system consists of two redundant independent trains that monitor all operable core exit thermocouples (each train monitoring up to 25 channels). A layout sketch of the system is shown in figure 4A.1. The core exit thermocouples are mounted at the top of the core support plate. They are then routed to four upper head core exit thermocouple nozzle assemblies (CETNA) penetrations. After exiting the CETNA penetrations, the thermocouple wires proceed through a swagelok and then to qualified connectors to facilitate disconnection during removal of the upper head. Upon exiting the reactor vessel cavity, the cables are routed in a manner consistent with the requirements of Regulatory Guide 1.75 to the in-containment qualified reference junction boxes. Each reference junction box includes three redundant platinum resistance temperature detectors (RTDs) imbedded in a block of copper to reflect the temperature at the junction of the chromel alumel to copper wire. The uncompensated core exit thermocouple signals (up to 25 per train) and the reference junction box temperatures (3) are routed to remote processing units (RPUs) A3 and B3. The signals from both RPUs are routed to both display processing units (DPUs) for calculation of the compensated core exit thermocouple value. The value chosen for the reference junction box temperature is a function of the data quality of each of the RTD signals. Following the calculation of all operable compensated thermocouple values, the information from both DPUs are transmitted to both seismically qualified flat panel plant safety monitoring system (PSMS) displays. The displays are located on section D of the Plant Vogtle control board. (See figure 18.1-1.) DPU-A and display A are powered by train A and DPU-B, and display B are powered by train B. The cabling between the RPUs, DPUs, and displays meets the requirements of Regulatory Guide 1.75.

B. Core Subcooling Margin Monitor

The inputs to the core subcooling margin monitor include the following:

- Wide range reactor coolant system (RCS) pressure (4 channels)
- Core exit compensated thermocouple values (50 channels, excluding invalid channels)

Reference junction box RTD values (6 channels)

The electrical layout of the subcooling margin monitor is shown in figure 4A-2. One channel of wide range RCS pressure is input into each RPU channel (A2, A3, B2, and B3). Also the uncompensated thermocouple channels and the corresponding three reference junction box RTD signals are input into RPUs A3 and B3. The outputs of each of the RPUs are routed to each DPU. The RCS subcooling margin is then calculated based upon the wide range RCS pressure and compensated core exit thermocouple readings. The value of RCS pressure utilized in the calculation is a function of the quality of the pressure readings. The value of core exit thermocouple temperature is based upon the auctimeered high quadrant thermocouple average reading. The subcooling margin calculated values are routed to both displays (A and B). The cable routing from sensor input to display meet the requirements of Regulatory Guide 1.75. The PSMS displays are the same display panels utilized in displaying the core exit thermocouple information.

C. Reactor Vessel Level Instrumentation System

The reactor vessel level instrumentation system (RVLIS) consists of two redundant independent trains that monitor the water level in the reactor vessel.

The wide range RVLIS reading provides an indication of reactor vessel water level from the bottom of the vessel to the top of the vessel during natural circulation conditions. The narrow range RVLIS reading provides an indication of reactor vessel water level from the middle of the hot leg pipe to the top of the reactor vessel head during natural circulation conditions. The dynamic head RVLIS reading provides an indication of reactor core, internals, and outlet nozzle pressure drop for any combination of operating reactor coolant pumps. Comparison of the measured pressure drop with the normal, single phase pressure drop provides an approximate indication of the relative void content of the circulating fluid. The inputs to the RVLIS system include the following:

- 1. RCS hot leg wide range RTD's (2 channels)
- 2. Wide range RCS pressure (4 channels)
- 3. Differential pressure (6 channels)
- 4. Reference leg temperature values (14 channels)
- 5. Reactor coolant pump status (4 channels)

A fluid diagram of one train of the VEGP RVLIS system is shown in figure 4A-3 for the inputs associated solely with the RVLIS system. The electrical block diagram associated with the RVLIS system is shown in figure 4A-4.

As discussed, the RCS hot leg wide range RTD signals are input to RPUs A2 and B1. Also, one wide range RCS pressure channel is input into each RPU (A2, A3, B2, and B3).

In addition, one of two sets of three differential pressure signals (wide range, narrow range, and dynamic head) are input into RPU A3 and B3, respectively. Also seven reference leg compensating temperature inputs from each train of RVLIS are input into RPUs A3 and B3. Finally, to determine the appropriate RVLIS indication, the running status of each reactor coolant pump is input into the non-1E RPU N1.

- 4A.2 Several analyses have been performed to verify the design of the RVLIS system described in item 4a.1c. The results of these are discussed in the following documents:
 - A. Summary Report, Westinghouse Reactor Vessel Level Instrumentation System for Monitoring Inadequate Core Cooling, December 1980, submitted to the NRC via T. M. Anderson to Darrell G. Eisenhut, NS-TMA-2358 dated December 23, 1980.
 - B. Responses to NRC Request for Additional Information on the Westinghouse RVLIS, Summary Report.
 - C. Supplemental Information on the Westinghouse RVLIS, submitted to the NRC via E. P. Rahe to L. E. Phillips, NS-EPR-2579 dated March 19, 1982.

In addition to the analyses conducted in the three references above, the hydraulic components of the RVLIS system were installed at the Semiscale Test Facility in Idaho so that transient response characteristics could be obtained during small-break loss-of-coolant accident (LOCA) and other accident conditions. A description of the tests conducted and a discussion of the test results are presented in the following documents:

- D. Westinghouse Evaluation of RVLIS Performance at the Semiscale Test Facility, December 1981, submitted to the NRC via E. P. Rahe to L. E. Phillips, NS-EPR-2526 dated December 8, 1981.
- E. Westinghouse Evaluation of RVLIS Performance at the Semiscale Test Facility for Test S-UT-8, January 1982, submitted to the NRC via E. P. Rahe to L. E. Phillips, NS-EPR-2542 dated January 13, 1982.
- F. Westinghouse Evaluation of RVLIS performance at the Semiscale Test Facility for Test S-IB-7 submitted to the NRC via E. P. Rahe to L. E. Phillips, SED-SA-00081 dated June 28, 1982.
- 4A.3 A description of the tests conducted on the Westinghouse RVLIS system and the results of the tests are presented in references D, E, and F listed above.

Hardware (from sensor to computer inputs) similar to that installed on VEGP is currently functioning at several operating plants for monitoring inadequate core cooling. The algorithms for computing the core exit temperature, core subcooling margin, and reactor vessel level utilized in hardware at the operating plants is similar to that implemented at VEGP.

4A.4 Response to II.F.2, Attachment I,^(a) Design and Qualification Criteria for Pressurized Water Reactor Incore Thermocouples

- A. Attachment I provides design of the display package on the PSMS. The display package hierarchy, as summarized from Attachment I, includes the following:
 - 1. Top level plant status summary

4A-3

^a Westinghouse copyrighted 1985, not included as part of the FSAR.

- 2. Four lower level graphic displays
 - a. Core temperature map
 - b. Pressure-temperature operating limits
 - c. Reactor vessel water level
 - d. Nuclear power
- 3. Four pages of menu display
 - a. Primary Data Trend Menu
 - b. Secondary Data Trend Menu
 - c. Containment Data Trend Menu
 - d. Detailed Data Menu
- 4. Four multi-page sets of data
 - a. Six-page set of primary data trends
 - b. Five-page set of secondary data trends
 - c. Two-page set of containment data trends
 - d. Eight-page set of detailed data
- B. The following provides a top down display of the core exit thermocouple information:
 - 1. a. Maximum core exit thermocouple temperature.
 - b. Quadrant core exit thermocouple maximum, average and maximum temperature. Also provides a comparison between the RCS hot leg RTDs and the quadrant T/C data.
 - c. Spatially oriented core exit thermocouple map showing each thermocouple temperature.
 - d. Alphanumeric listing of core exit thermocouple location, tag designation, and temperature reading per quadrant.
 - e. A 2-h trend history of the four core exit thermocouple quadrant maximum temperatures.
 - 2. The core exit thermocouple display pages are designed such that any numeric thermocouple readout greater than 1200 F will be flashed at a frequency of 1 hertz.
- C. The following provides a summary of the top down display of the core subcooling margin (based upon core exit thermocouples):

- 1. a. Core subcooling margin based upon core exit thermocouples.
 - b. RCS pressure-temperature plot exhibiting plant approach to saturation.
 - c. Alphanumeric listing of both trains of core subcooling margin.
 - d. A 2-h trend history of the core subcooling margin.
- 2. The core subcooling margin will indicate "SUBCOOL" when the maximum core exit thermocouple temperature is at or below the RCS coolant saturation point. "SUPERHEAT" and the appropriate numeric value in degrees F will be displayed in reverse video when the maximum core exit thermocouple temperature exceeds the coolant saturation temperature.
- D. The following provides a summary of the top down display of the RVLIS system.
 - 1. Displays appropriate RVLIS narrow and wide range and dynamic head readings depending upon RCP status.
 - 2. Mimic of analog meters indicating RVLIS narrow, wide, and dynamic readings with respect to reactor vessel. Only displays appropriate ranges based upon RCP status.
 - 3. Alphanumeric listing of appropriate ranges for both trains of RVLIS system.
 - 4. A 2-h trend history of all three RVLIS ranges. Also presents a trend of RCP status.
- E. Since the VEGP PSMS display system features two redundant independent displays, one display console is considered the primary display and the other display console is considered the backup display. As such, the backup display console for ICC monitoring is also a qualified display.
- F. The ranges of the ICC instrumentation are given in table 7.5.2-1.

4A.5 Response to II.F.2, Appendix B, Design and Qualification Criteria for Accident Monitoring Instrumentation

- A. Equipment Qualification
 - 1. Core Exit Thermocouple Monitoring

Listed below are the appropriate documents indicating the qualification tests conducted on the PSMS subsystems.

Subsystem	<u>Document</u>
Al-Ch Connectors	ESE-43B,C
Reference junction box	ESE-44A
Microprocessors	ESE-53

Plasma display ESE-63B

2. Core Subcooling Margin Monitoring

<u>Subsystem</u> <u>Document</u>

Wide range RCS pressure ESE-2

Core exit thermocouples See item above

Microprocessors ESE-53

Plasma display ESE-63B

3. RVLIS Monitoring System

<u>Subsystem</u> <u>Document</u>

Wide range RCS pressure ESE-1A

Differential pressure ESE-4

Wide range RTDs ESE-6

High volume pressure sensor ESE-48A

Hydraulic isolator ESE-49A

Reference leg RTDs ESE-42A

Microprocessors ESE-53

Plasma display ESE-63B

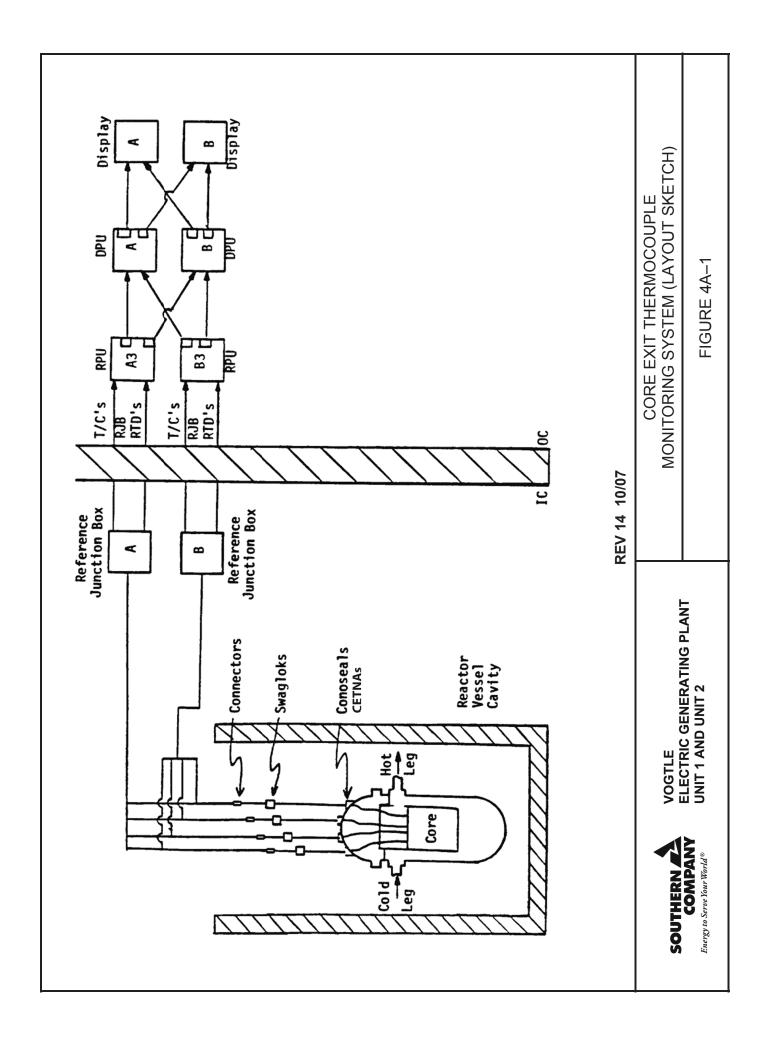
B. Single Failure Criteria

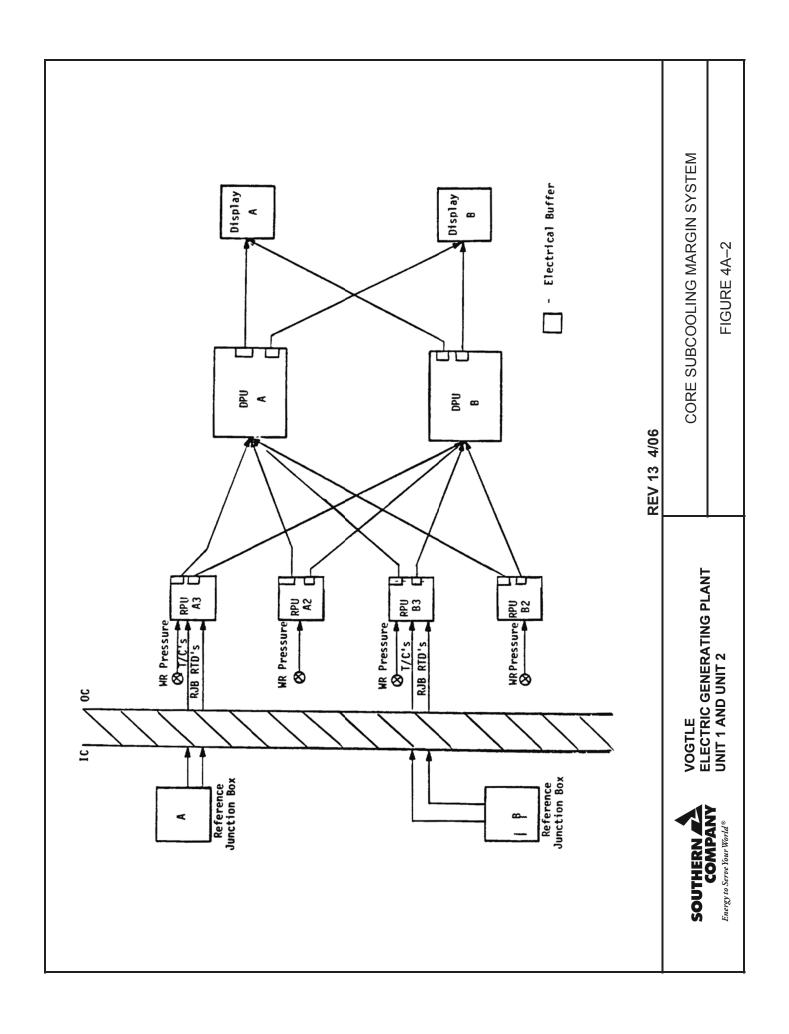
A detailed discussion of the Regulatory Guide 1.97, Post Accident Monitoring Design Basis, is presented in section 7.5 of the VEGP FSAR. Included in the discussion is a justification for the number of channels selected and the diverse variable identified where necessary. Discussed in FSAR section 7.5 is a detailed description of the characteristics associated with each ICC monitoring system.

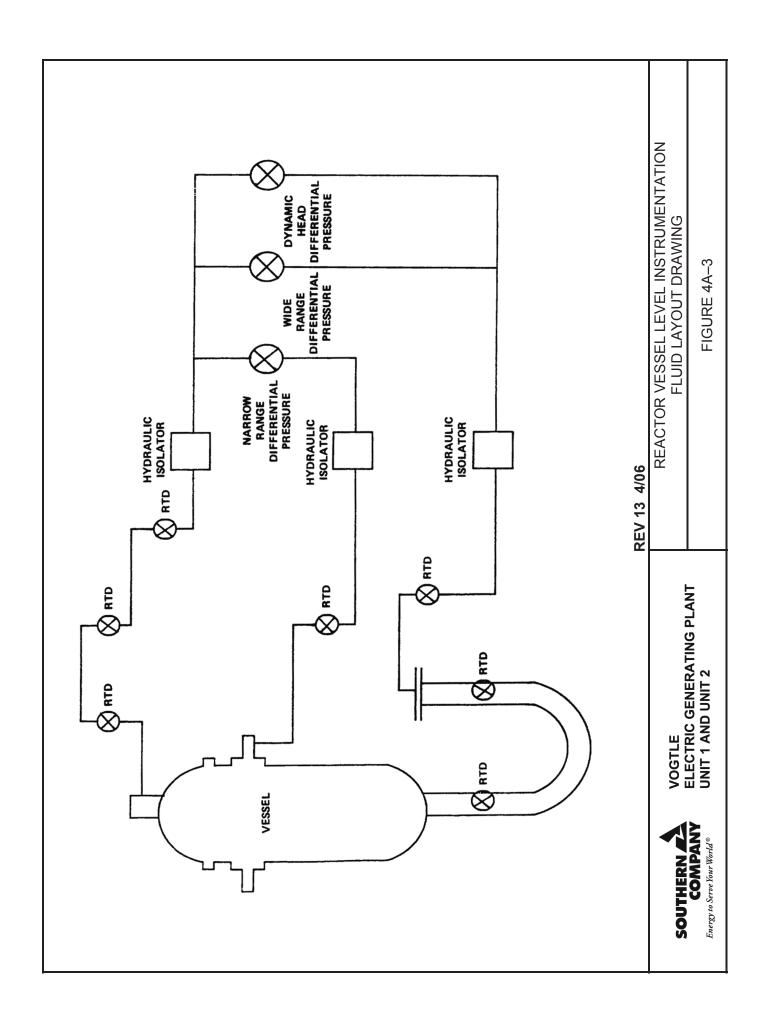
C. RPUs A1 and A2, DPU-A, and Display A are provided by inverter power bus I. RPUs B1 and B2, DPU-B, and display B are powered by inverter power bus II. RPU A3 is powered by inverter power bus III, and RPU B3 is powered by inverter power bus IV.

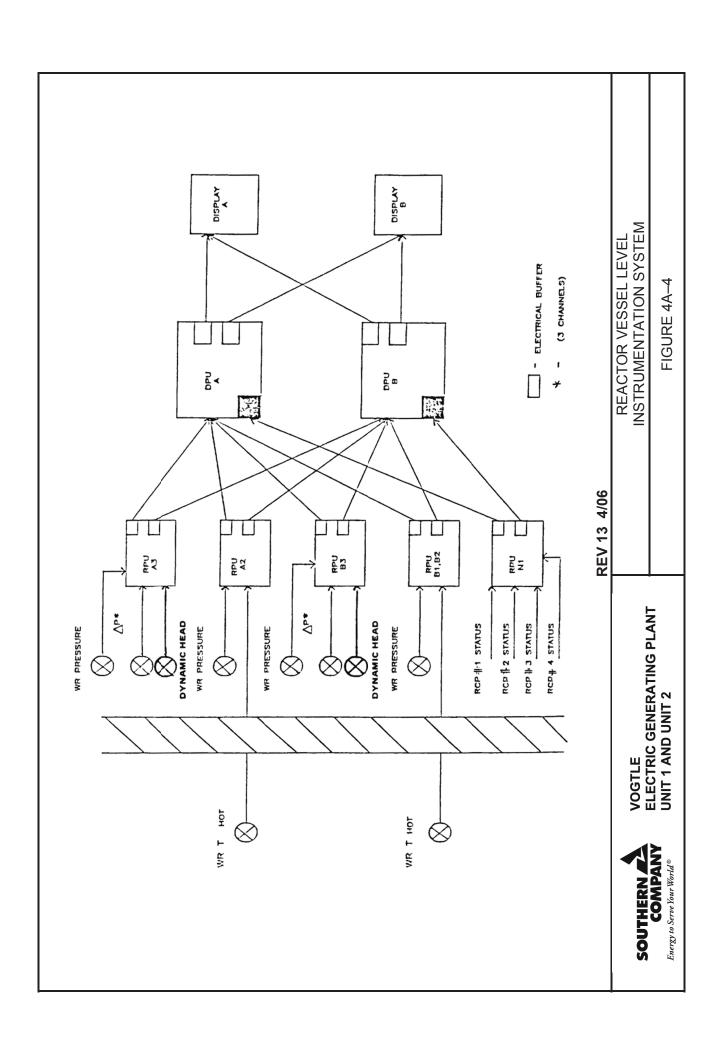
A sketch of signal flows between the protection channels, RPUs, DPUs, and displays is shown in figure 4A-5.

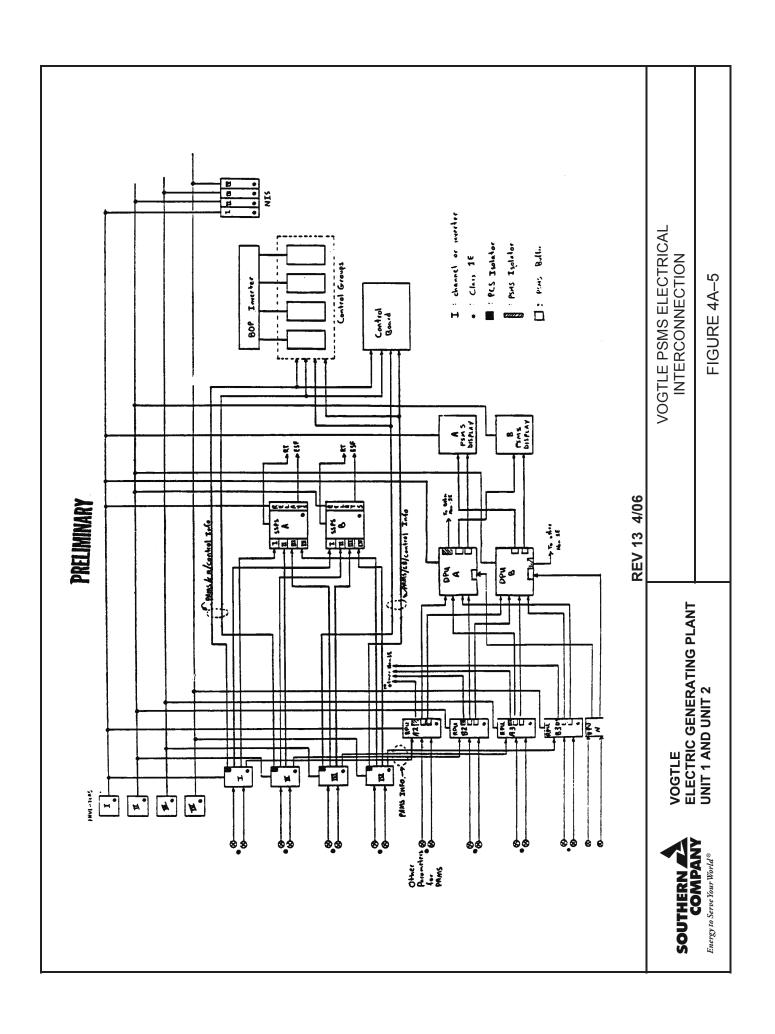
- 4A.6 VEGP is adopting the format and content of the Westinghouse Owners Group (WOG) Emergency Response Guidelines for writing the plant specific procedures. Attachment II illustrates the generic WOG. These procedures provide for a critical safety function status tree for monitoring the status of plant core cooling. All variables necessary to implement the core cooling status tree are provided by the VEGP ICC instrumentation system. The functional restoration guideline, to which the operator is directed based upon the logic dictated by the tree, also utilizes the information provided by the ICC instrumentation.
- 4A.7 Evaluation of the acceptability of the location of the plant safety monitoring system (PSMS) displays will be included as part of the detailed control room design review.











5.0 REACTOR COOLANT SYSTEM AND CONNECTED SYSTEMS

5.1 SUMMARY DESCRIPTION

This section describes the reactor coolant system (RCS) and includes a schematic flow diagram (figure 5.1.2-1), a piping and instrumentation diagram (drawings 1X4DB111, 2X4DB111, 1X4DB112, 1X4DB113, and 2X4DB113), and an elevation drawing (drawings 1X4DL4A17 and 2X4DL4A17).

5.1.1 DESIGN BASES

The performance and safety design bases of the RCS and its major components are interrelated. These design bases are listed below:

- A. The RCS has the capability to transfer to the steam and power conversion system the heat produced during power operation and when the reactor is subcritical, including the initial phase of plant cooldown.
- B. The RCS has the capability to transfer to the residual heat removal system the heat produced during the subsequent phase of plant cooldown and cold shutdown.
- C. The RCS heat removal capability under power operation and normal operational transients, including the transition from forced to natural circulation, ensures no fuel damage within the operating bounds permitted by the reactor control and protection systems.
- D. The RCS provides the water used as the core neutron moderator and reflector and as a solvent for the neutron absorber used as chemical shim control.
- E. The RCS maintains the homogeneity of the soluble neutron poison concentration and the rate of change of the coolant temperature so that uncontrolled reactivity changes do not occur.
- F. The RCS pressure boundary is capable of accommodating the temperatures and pressures associated with operational transients.
- G. The reactor vessel supports the reactor core and control rod drive mechanisms.
- H. The pressurizer maintains the system pressure during operation and limits pressure transients. During the reduction or increase of plant load, the pressurizer accommodates volume changes in the reactor coolant.
- I. The reactor coolant pumps supply the coolant flow necessary to remove heat from the reactor core and transfer it to the steam generators.
- J. The steam generators provide high-quality steam to the turbine. The tube and tube sheet boundary are designed to prevent the transfer of radioactivity generated within the core to the secondary system.

- K. The RCS piping contains the coolant under operating temperature and pressure conditions and limits leakage (and activity release) to the containment atmosphere. The RCS piping contains demineralized borated water that is circulated at the flowrate and temperature consistent with achieving the reactor core thermal and hydraulic performance.
- L. The RCS is monitored for loose parts, as described in subsection 4.4.6.
- M. Applicable industry standards and equipment classifications of RCS components are identified in table 3.2.2-1.
- N. The reactor vessel is provided with a head vent that meets the requirements of TMI Action Item II.B.1. (See subsections 5.4.7 and 5.4.15.)
- O. Unisolable sections of safety injection, normal and alternate charging, and auxiliary spray lines interconnected with the reactor coolant system, two 12-in. residual heat removal suction lines attached to the reactor coolant loop, and the pressurizer surge line are instrumented with resistance temperature detectors (RTDs) strapped on the pipe to detect thermal stratification. (See paragraph 5.4.3.3.4.)

5.1.2 DESIGN DESCRIPTION

The reactor coolant system (RCS), shown in drawings 1X4DB111, 2X4DB111, 1X4DB112, 2X4DB112, and 1X4DB113, consists of four similar heat transfer loops connected in parallel to the reactor pressure vessel. Each loop contains a reactor coolant pump, steam generator, and associated piping and valves. In addition, the system includes a pressurizer, pressurizer relief and safety valves, interconnecting piping, and instrumentation necessary for operational control. All the above components are located in the containment building.

During operation, the RCS transfers the heat generated in the core to the steam generators, where steam is produced to drive the turbine-generator. Borated demineralized water is circulated in the RCS at a flowrate and temperature consistent with achieving the reactor core thermal-hydraulic performance. The water also acts as a neutron moderator and reflector and as a solvent for the neutron absorber used in chemical shim control.

The RCS pressure boundary provides a barrier against the release of radioactivity generated within the reactor and is designed to ensure a high degree of integrity throughout the life of the plant.

The RCS pressure is controlled by the use of the pressurizer where water and steam are maintained at saturation conditions by electrical heaters and water sprays. Steam can be formed (by the heaters) or condensed (by the pressurizer spray) to minimize pressure variations due to contraction and expansion of the reactor coolant. Spring-loaded safety valves and power-operated relief valves connected to the pressurizer provide for steam discharge from the RCS. Discharged steam is piped to the pressurizer relief tank (pressurizer relief discharge system), where the steam is condensed and cooled by mixing with water.

The extent of the RCS is defined as:

- The reactor vessel, including control rod drive mechanism housings.
- The portion of the steam generators containing reactor coolant.
- The reactor coolant pumps.

- The pressurizer.
- The safety and relief valves.
- The head vent.
- The interconnecting piping, valves, and fittings between the principal components listed above.
- The piping, fittings, and valves leading to connecting auxiliary or support systems.

The RCS schematic flow diagram is shown in figure 5.1.2-1. Included with this figure is a tabulation of principal pressures, temperatures, and flowrates of the system under normal steady-state, full-power operating conditions. These parameters are based on the best-estimate flow at the pump discharge. The RCS volume under these conditions is presented in table 5.1.2-1.

A piping and instrumentation diagram of the RCS is shown in drawings 1X4DB111, 2X4DB111, 1X4DB112, 2X4DB112, and 1X4DB113. This diagram shows the extent of the systems located within the containment and the points of separation between the RCS and the secondary (heat utilization) system. Drawings 1X4DE312, 1X4DE313, 1X4DE314, 1X4DE317, 1X4DE320, 1X4DE322, 1X2D48E007 and 1X2D48E008 provide plan and elevation views of the containment, and drawings 1X4DL4A17 and 2X4DL4A17 show plan and section of the reactor coolant loops. These figures show principal dimensions of RCS components in relationship to supporting and surrounding steel and concrete structures and demonstrate the protection provided to the RCS by its physical layout.

5.1.3 SYSTEM COMPONENTS

The major components of the reactor coolant system are as follows:

A. Reactor Vessel

The reactor vessel is cylindrical and has a welded, hemispherical bottom head and a removable, flanged, hemispherical upper head. The vessel contains the core, core support structures, control rods, and other parts directly associated with the core.

The vessel has inlet and outlet nozzles located in a horizontal plane just below the reactor vessel flange but above the top of the core. Coolant enters the vessel through the inlet nozzles and flows down the core barrel-vessel wall annulus, turns at the bottom, and flows up through the core to the outlet nozzles.

B. Steam Generators

The steam generators are vertical shell and U-tube evaporators with integral moisture-separating equipment. The reactor coolant flows through the inverted U-tubes, entering and leaving through the nozzles located in the hemispherical bottom head of the steam generator. Steam is generated on the shell side and flows upward through the moisture separators to the outlet nozzle at the top of the vessel.

C. Reactor Coolant Pumps

The reactor coolant pumps (RCPs) are single-speed centrifugal units driven by water/air-cooled, three-phase induction motors. The shaft is vertical with the motor mounted above the pump. A flywheel on the shaft above the motor provides additional inertia to extend pump coastdown. The flow inlet is at the bottom of the pump, and the discharge is on the side.

D. Piping

The reactor coolant piping is seamless, stainless steel piping. The hot leg is defined as the piping between the reactor vessel outlet nozzle and the steam generator. The mechanical stress improvement process (MSIP) has been applied to the Unit 1 and Unit 2 hot legs near the reactor vessel outlet nozzle. The cold leg is defined as the piping between the RCP outlet and the reactor vessel. The crossover leg is defined as the piping between the steam generator outlet and the RCP inlet.

E. Pressurizer

The pressurizer is a vertical, cylindrical vessel with hemispherical top and bottom heads. The pressurizer is connected to the hot leg of one of the coolant loops by a surge line. Electrical heaters are installed through the bottom head of the vessel. The spray nozzle and relief and safety valve connections are located in the top head of the vessel.

F. Safety and Relief Valves

The pressurizer safety valves are of the totally enclosed pop type. The valves are spring loaded and self-activated with backpressure compensation. The power-operated relief valves are solenoid-operated valves. They are operated automatically or by remote manual control. Remotely operated gate valves are provided to isolate the inlet to the power-operated relief valves if excessive leakage occurs. Position-indicating lights are provided in the control room for these valves.

5.1.4 SYSTEM PERFORMANCE CHARACTERISTICS

Design and performance characteristics of the reactor coolant system are provided in table 5.1.2-1.

A. Reactor Coolant Flow

The reactor coolant flow, a major parameter in the design of the system and its components, is established by a detailed design procedure supported by operating plant performance data and component hydraulics experimental data. The procedure establishes a best-estimate flow as well as conservatively high and low flows for the applicable mechanical and thermal design considerations. In establishing the range of design flows, the procedure accounts for the uncertainties in the component flow resistances and the pump head-flow capability, established by analysis of the available experimental data. The procedure also accounts for the uncertainties in the technique used to measure flow in the operating plant.

Definitions of the three reactor coolant flows applied in various plant design considerations are presented in the following paragraphs.

B. Best-Estimate Flow

The best-estimate flow is considered to be the most likely value for the plant operating condition. This flow is based on the best estimate of the reactor vessel, steam generator, and piping flow resistances and on the best estimate of the reactor coolant pump (RCP) head-flow capability, with no known uncertainties assigned to either the system flow resistance or the pump head. The best-estimate flow provides the basis for the establishment of the other design flows required for the system and component design. System pressure losses based on best-estimate flow are presented in table 5.1.2-1.

The best-estimate flow analysis has been based on extensive experimental data, including accurate flow and pressure drop data from one operating plant, flow resistance measurements from several fuel assembly hydraulics tests, and hydraulic performance measurements from several pump impeller model tests. Since operating plant flow measurements have been shown to be in close agreement with the calculated best-estimate flows, the flows established with this design procedure can be applied to the plant design with a high level of confidence.

Although the best-estimate flow is the most likely value to be expected in operation, more conservative flowrates are applied in the thermal and mechanical designs.

C. Thermal Design Flow

Thermal design flow is the flowrate used as a basis for the reactor core thermal performance, the steam generator thermal performance, and the nominal plant parameters used throughout the design. The thermal design flow accounts for the uncertainties in flow resistances (reactor vessel, steam generator, and piping), RCP head, and the methods used to measure flowrate. The thermal design flow is approximately 5.8% less than the best-estimate flow with 10% equivalent steam generator plugging. The thermal design flow is confirmed when the plant is placed in operation. Tabulations of important design parameters based on the thermal design flow are provided in table 5.1.2-1.

D. Mechanical Design Flow

Mechanical design flow is a conservatively high flow used in the mechanical design of the reactor vessel internals and fuel assemblies. The mechanical design flow is based on a reduced system resistance and on increased pump head capability. The mechanical design flow is approximately 1.7% greater than the best-estimate flow with 0% equivalent steam generator tube plugging.

Pump overspeed due to a turbine generator overspeed of 20% results in a peak reactor coolant flow of 120% of the mechanical design flow. The overspeed condition is applicable only to operating conditions when the reactor and turbine generator are at power.

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TABLE 5.1.2-1 (SHEET 1 OF 2)

SYSTEM DESIGN AND OPERATING PARAMETERS

Plant design life (years)	40 ^g	
Nominal operating pressure (psig)	2235	
Total system volume, including pressurizer and surge line (ft ³)	12,347 ^(a)	
System liquid volume, including pressurizer water at maximum guaranteed power (ft³)	11,594 ^(a)	
Pressurizer spray rate, maximum (gal/min)	900	
Pressurizer heater capacity (kW)	1800 / 1661 (Unit 2)	
Minimum pressurizer heater capacity (kW) required to survive 3/8-in. line break w/o Rx trip or ECCS actuation (does not reflect 100 kW heat loss reduction)	1624	
System Thermal and Hydraulic Data	4 Pumps Running (b)	(e)
Nuclear steam supply system power (MW)	3643	
Reactor power (MW)	3626	
Thermal design flows (gal/min) Active loop Reactor	93,600 374,400	
Total reactor flow (10 ⁶ lb/h)	139.45	143.1
Temperatures (°F) Reactor vessel outlet	620.6	603.8
Reactor vessel inlet	556.2	537.6
Steam generator outlet	555.9	537.3
Steam generator steam	537.3	520.6
Feedwater	448.7	448.7
Steam pressure (psia)	941	817

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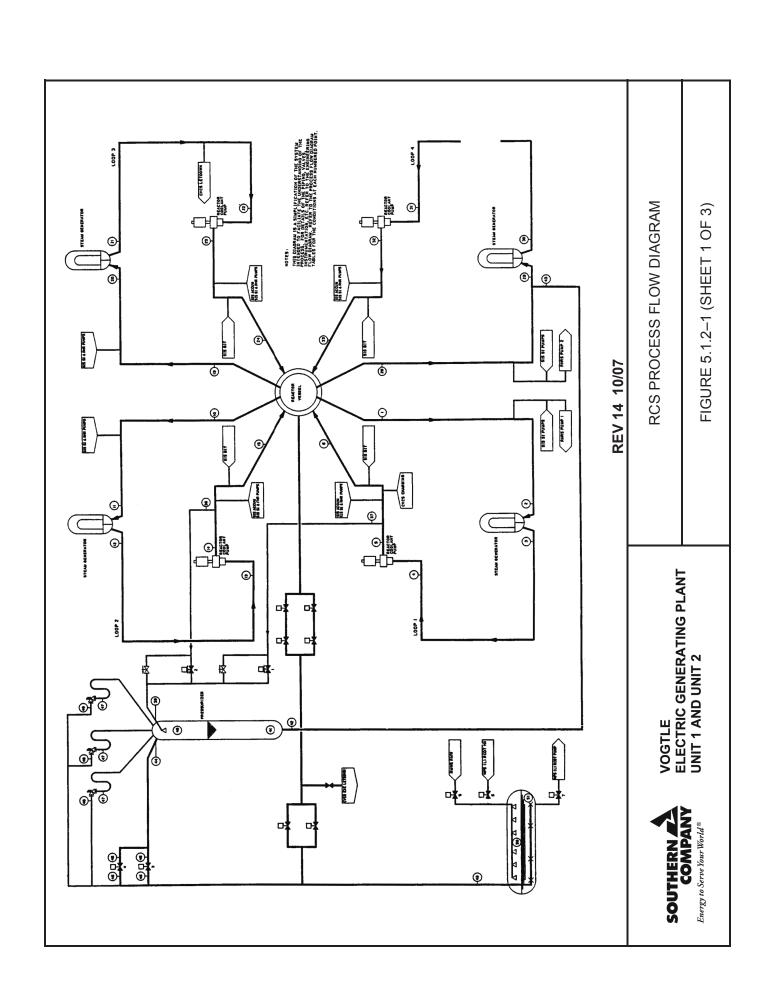
TABLE 5.1.2-1 (SHEET 2 OF 2)

Total steamflow (10 ⁶ lb/h)	16.31	16.22
Best-estimate flows (gal/min) Active loop Reactor	99,400 397,600	102,500 410,000
Mechanical design flows (gal/min) Active loop Reactor	104,200 416,800	
System pressure drops ^(f) Reactor vessel ΔP (psi) Steam generator ΔP (psi) Hot leg piping ΔP (psi) Crossover leg piping ΔP (psi)	46.5 45.5 1.2 3.1	47.8 41.3 1.3 3.4
Crossover leg piping ΔP (psi) Cold leg piping ΔP (psi) Pump head (ft)	3.3 ^(d) 309	3.6 ^(d) 295

- c. Includes core, internals, and nozzles.
- d. Includes pump weir ΔP of 2.0 psi.
- e. Parameter based on full power operation with 0% equivalent steam generator tube plugging and reactor vessel average temperature of 570.7.
- f. System pressure drops are based on best estimate flow.
- g. The operating licenses for both VEGP units have been renewed and the original licensed operating terms have been extended by 20 years, resulting in a plant operating life of 60 years. In accordance with 10 CFR Part 54, appropriate aging management programs and activities have been initiated to manage the detrimental effects of aging to maintain functionality during the period of extended operation (see chapter 19).

a. Nominal volumes reflecting 0% equivalent steam generator tube plugging.

b. Parameters based on full power operation with 10% equivalent steam generator tube plugging and reactor vessel average temperature of 588.4°F.



Reactor Reactor Coolant Reactor Reactor Reactor Coolant H Reactor	ı	iemperature 140			
	2246.6		1391/1111	LIDINIO	(121)
		618.2	111,570	37.0430	ı
	2245.2	618.2	111,680	37.0783	•
	2206.0	558.6	99,930	37.0783	r
	2202.9	558.6	100, 100	37.1397	•
5 Reactor coolant	2295.6	558.8	086,980	37.1372	1
6 Reactor coolant	2292.3	558.8	99,910	37.1086	ı
10-16 Reactor coolant	See loop No. 1 specifications.	pecifications.			
19-24 Reactor coolant	See loop No. 1 specifications.	pecifications.			
28-33 Reactor coolant	See loop No. 1 specifications.	pecifications.			
37 Reactor coolant	2296.5	558.8	1.0	0.00037	1
38 Reactor coolant	2295.5	558.8	1.0	0.00037	•
39 Reactor coolant	2235.0	652.7	1.0	0.00074	•
40 Steam	2235.0	652.7	1	1	720
	Œ.	REV 14 10/07			

VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2

SOUTHERN COMPANY
COMPANY
Energy to Serve Your World®

RCS PROCESS FLOW DIAGRAM

FIGURE 5.1.2-1 (SHEET 2 OF 3)

Volume (ft3)	1080	•	Minimize	Minimize		Minimize			450	1350
(1b/h×10)		0.00074	0	0	0	0	0	0	0	ı
(gai/min) (1b/hx10)	•	2.50	0	0	0	0	0	0	0	1
Nominal Temperature	652.7	652.7	652.7	652.7	120	652.7	120	120	120	120
Pressure (psiq)	2244.0	2244.0	2235.0	2235.0	3.0	2235.0	3.0	3.0	3.0	3.0
Fluid	Reactor	Reactor	Steam	Reactor coolant	N 2	Reactor	N ₂	N ₂	N ₂ .	Pres- surizer relief tank water
Location	5	42	ħħ	45	917	47	84	64	20	51

b. At the conditions specified.



VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2

RCS PROCESS FLOW DIAGRAM

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FIGURE 5.1.2-1 (SHEET 3 OF 3)

Simplifying assumption is made that letdown and charging flows are distributed over all loops. Charging, letdown, and continuous spray flows are ignored in overall balance. φ.

5.2 INTEGRITY OF REACTOR COOLANT PRESSURE BOUNDARY

This section discusses the measures employed to provide and maintain the integrity of the reactor coolant pressure boundary (RCPB) for the plant design lifetime^a. Section 50.2 of 10 CFR 50 defines the RCPB as extending to the outermost containment isolation valve in system piping which penetrates the containment and is connected to the reactor coolant system (RCS). This section is limited to a description of the components of the RCS, as defined in section 5.1, unless otherwise noted. Components which are part of the RCPB (as defined in 10 CFR 50) but are not described in this section are described in the following sections:

- A. Section 6.3 RCPB components which are part of the emergency core cooling system.
- B. Subsection 9.3.4 RCPB components which are part of the chemical and volume control system.
- C. Subsection 3.9.N.1 Design loading, stress limits, and analyses applied to the RCS and American Society of Mechanical Engineers (ASME) Code Class 1 components.
- D. Subsection 3.9.N.3 Design loadings, stress limits, and analyses applied to ASME Code Class 2 and 3 components.

The abbreviation RCS, as used in this section, is as defined in section 5.1. When the term RCPB is used in this section, its definition is that of Section 50.2 of 10 CFR 50.

5.2.1 COMPLIANCE WITH CODES AND CODE CASES

5.2.1.1 <u>Compliance with 10 CFR 50.55a</u>

RCS components are designed and fabricated in accordance with 10 CFR 50, Section 50.55a, Codes and Standards. The addenda of the ASME code applied in the design of each component are listed in table 5.2.1-1.

5.2.1.2 Applicable Code Cases

Regulatory Guides 1.84 and 1.85 are discussed in section 1.9. The following discussion addresses only unapproved or conditionally approved code cases (per Regulatory Guides 1.84 and 1.85) used on Class 1 components.

Code Case 1528 (SA-508, Class 2a) material has been used in the manufacture of the VEGP steam generators and pressurizers. Regulatory Guide 1.85 presently reflects a conditional Nuclear Regulatory Commission (NRC) approval of Code Case 1528. Westinghouse has conducted a test program which demonstrates the adequacy of Code Case 1528 material. The

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^a The operating licenses for both VEGP units have been renewed and the original licensed operating terms have been extended by 20 years. In accordance with 10 CFR Part 54, appropriate aging management programs and activities have been initiated to manage the detrimental effects of aging to maintain functionality during the period of extended operation (see chapter 19).

results of the test program are documented in reference 1. Reference 1 and a request for approval (reference 2) of the use of Code Case 1528 have been submitted to the NRC.

5.2.1.3 References

- 1. Letter NS-CE-1228, dated October 4, 1976, C. Eicheldinger (Westinghouse) to J. F. Stolz (NCR).
- 2. Letter NS-CE-173, dated March 17, 1978, C. Eicheldinger (Westinghouse) to J. F. Stolz (NRC).

5.2.2 OVERPRESSURE PROTECTION

Reactor coolant system (RCS) overpressure protection is provided by the pressurizer safety valves, steam generator safety valves, and the reactor protection system and associated equipment. Combinations of these systems ensure compliance with the overpressure requirements of the American Society of Mechanical Engineers (ASME), Boiler and Pressure Vessel Code, Section III, Paragraphs NB-7300 and NC-7300, for pressurized water reactor systems.

The only portions of an auxiliary system connected to the RCS that are utilized for overpressure protection of the RCS are the liquid relief valves of the residual heat removal system (RHRS). These valves protect the RCS at low temperatures when the RHRS is in operation.

5.2.2.1 Design Bases

Overpressure protection is provided for the RCS by the pressurizer safety valves. This protection is afforded for the following events which envelope those credible events that could lead to overpressure of the RCS if adequate overpressure protection is not provided:

- Loss of electrical load and/or turbine trip.
- Uncontrolled rod withdrawal at power.
- Loss of reactor coolant flow.
- Loss of normal feedwater.
- Loss of offsite power to the station auxiliaries.

The sizing of the pressurizer safety valves is based on analysis of a complete loss of steamflow to the turbine with the reactor operating at 102 percent of engineered safeguards design power. In this analysis, feedwater flow is also assumed to be lost, and no credit is taken for operation of the pressurizer power-operated relief valves (PORV), pressurizer level control system, pressurizer spray system, rod control system, steam dump system, or steam line PORV. The reactor is maintained at full power (no credit for direct reactor trip on turbine trip), and steam relief through the steam generator safety valves is considered. The total pressurizer safety valve capacity is required to be at least as large as the maximum surge rate into the pressurizer during this transient.

This sizing procedure results in a safety valve capacity well in excess of the capacity required to prevent exceeding 110 percent of system design pressure for the events listed above.

Overpressure protection for the steam system is provided by steam generator safety valves. The steam system safety valve capacity is based on providing enough relief to remove the engineered safeguards design steamflow. This must be done while limiting the maximum steam system pressure to less than 110 percent of the steam generator shell side design pressure.

Blowdown and heat dissipation systems of the nuclear steam supply system (NSSS) connected to the discharge of these pressure relieving devices are discussed in subsection 5.4.11.

Steam generator blowdown systems are discussed in subsection 10.4.8.

5.2.2.2 Design Evaluation

A description of the pressurizer safety valves performance characteristics along with the design description of the incidents, assumptions made, method of analysis, and conclusions are discussed in chapter 15.

The relief capacities of the pressurizer and steam generator safety valves are determined from the postulated overpressure transient conditions in conjunction with the action of the reactor protection system. An evaluation of the functional design of the overpressure protection system and an analysis of the capability of the system to perform its function for a typical plant are presented in reference 1. The report describes in detail the types and number of pressure relief devices employed, relief device description, locations in the systems, reliability history, and the details of the methods used for relief device sizing based on typical worst-case transient conditions and analysis data for each transient condition. An overpressure protection report specifically for the VEGP is prepared in accordance with Article NB-7300 of Section III of the ASME Code. The description of the analytical model used in the analysis of the overpressure protection system and the basis for its validity are discussed in reference 2.

The capacities of the pressurizer safety and relief valves are discussed in subsection 5.4.13. The setpoints and reactor trip signals which occur during overpressure transients are discussed in subsection 5.4.10.

5.2.2.3 Piping and Instrumentation Diagrams

Overpressure protection for the RCS is provided by the pressurizer safety and relief valves shown in drawing 1X4DB112. These valves discharge to the pressurizer relief tank through a common manifold.

The steam system safety valves are discussed in section 10.3 and are shown in drawings 1X4DB159-1, 1X4DB159-2, and 1X4DB159-3.

5.2.2.4 **Equipment and Component Description**

The operation, significant design parameters, number and types of operating cycles, and environmental conditions of the pressurizer safety valves are discussed in section 3.11.N and subsections 3.9.N.1 and 5.4.13.

Section 10.3 contains a discussion of the equipment and components of the steam system overpressure protection features.

5.2.2.5 <u>Mounting of Pressure Relief Devices</u>

The design and installation of the pressure relief devices for the RCS are described in subsection 5.4.11. The design basis for the assumed loads for the primary and secondary side pressure relief devices are described in section 3.9.N. Subsection 10.3.2 provides a discussion of the main steam safety valves and the power-operated atmospheric steam relief valves.

5.2.2.6 Applicable Codes and Classification

The requirements of the ASME Boiler and Pressure Vessel Code, Section III, Paragraphs NB-7300 (Overpressure Protection Report) and NC-7300 (Overpressure Protection Analysis), are met.

Piping, valves, and associated equipment used for overpressure protection are classified in accordance with American National Standards Institute (ANSI) N18.2a-1975, Nuclear Safety Criteria for the Design of Stationary Pressurized Water Reactor Plants. These safety class designations are delineated in table 3.2.2-1 and shown in drawings 1X4DB111, 2X4DB111, 1X4DB112, 2X4DB112, and 1X4DB113.

5.2.2.7 Material Specifications

Refer to subsection 5.2.3 for a description of material specifications.

5.2.2.8 Process Instrumentation

Each pressurizer safety valve discharge line incorporates a control board temperature indicator and alarm to notify the operator of steam discharge due to either leakage or actual valve operation. For a further discussion on process instrumentation associated with the system, refer to chapter 7.

5.2.2.9 System Reliability

The reliability of the pressure relieving devices is discussed in section 4 of reference 1.

5.2.2.10 RCS Pressure Control During Low-Temperature Operation^a

An important aspect of RCS overpressure protection at low temperatures is the use of administrative controls which are discussed in some detail in paragraph 5.2.2.10.2. Although specific alarms do not exist to invoke specific administrative procedures, annunciation is provided to alert the operator to arm the cold overpressure mitigation system. Operating procedures maximize the use of a pressurizer steam bubble, since a steam bubble reduces the maximum pressure reached for some transients, and slows the rate of pressure increase for others, and aids the operator in controlling RCS pressure during low temperature operation.

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^a The cold overpressure protection system setpoint calculation was evaluated as a time-limited aging analysis (TLAA) for license renewal in accordance with 10 CFR Part 54. The results of this evaluation are provided in paragraph 19.4.6.4.

When the RCS is at temperatures below approximately 350°F, it is opened to the RHRS for the purposes of removing residual heat from the core, providing a path for letdown to the purification subsystem, and controlling the RCS pressure when the plant is operating in a water solid mode. The RHRS is provided with self-actuated water relief valves to prevent overpressure in this relatively low design pressure system caused either within the system itself or from transients transmitted from the RCS. The RHRS relief valves mitigate pressure transients originated in the RCS to maximum pressure values determined by the relief valve set pressure.

The low design pressure RHRS is normally isolated from the high design pressure RCS during reactor power operation at temperatures above approximately 350°F by two isolation valves in series. Therefore, the RHRS can be inadvertently isolated from the RCS by these same isolation valves. The PORVs and associated logic provide overpressure mitigation for those transients which might occur if the RHRS isolation valves were inadvertently closed. The PORV logic is manually armed at the system setpoint.

Two pressurizer PORVs are each supplied with actuation logic. The logic for each PORV continuously monitors RCS temperature and pressure, converts an auctioneered RCS temperature to the Appendix G allowable pressure, and then compares the allowable pressure to the actual RCS pressure. As the actual RCS pressure approaches the allowable pressure, a main control board alarm is annunciated. If the RCS pressure continues to increase, an actuation signal is transmitted to a PORV and the valve opens to mitigate the transient.

As described in subsection 5.4.13, the VEGP PORVs are safety related. They were designed in accordance with the ASME Code and are qualified via the Westinghouse pump and valve operability program which is described in paragraph 3.10.N.2.2.

The hardware and logic associated with this function will operate following an operating basis earthquake. Offsite power is not required for the system to function. The actuation logic in the system is testable. However, the PORVs and RHR relief valves are not exercised with the reactor at power. They are capable of being tested as required by the ASME Code and the VEGP Technical Specifications.

5.2.2.10.1 Transient Evaluation

Potential overpressurization transients to the RCS, while at relatively low temperatures, can be caused by either of two types of events to the RCS; i.e., mass input or heat input. Both types result in more rapid pressure changes when the RCS is water solid.

Anticipated mass and heat input transients are evaluated to demonstrate conformance with Appendix G. The most limiting heat input transient is an inadvertant reactor coolant pump startup in a loop where the steam generator secondary temperature is 50°F higher than the primary temperature in any loop. The most limiting mass input transient is a charging-letdown mismatch where two emergency core cooling system (ECCS) centrifugal charging pumps and one normal centrifugal charging pump are charging water into the reactor coolant system with the letdown path isolated.

It should be noted that the following transient is also addressed. With the plant in a cooled down and depressurized condition in which the cold overpressure protection system is required to be operable and with charging and letdown established and RHRS open to RCS, a dc vital bus fails. This failure causes normal letdown to isolate and also results in the loss of one of the two PORVs. However, RHRS relief valves mitigate the transient.

5.2.2.10.2 Administrative Controls

During plant operation the following precautions are observed:

- A. At least one RHR inlet line from the reactor coolant loop is not isolated unless there is a steam bubble in the pressurizer. This precaution ensures that there is a relief path from the reactor coolant loop to a RHR suction line relief valve when the RCS is at low temperature and is water solid.
- B. Whenever the plant is water solid and the reactor coolant pressure is being maintained by the low pressure letdown control valve, letdown should include flow from the operating RHR loop through the RHRS cleanup to the letdown heat exchanger valve.
- C. One RCP should normally be running anytime RCS temperature is changed by more than 10°F in 1 h. Additionally, RCPs should not be started if steam generator secondary water temperature is greater than 10°F above the RCS temperature.
- D. During a typical plant cooldown, operable steam generators should be connected to the steam header to ensure a uniform cooldown of the reactor coolant loops.
- E. To preclude inadvertent emergency core cooling system (ECCS) actuation during heatup and cooldown, blocking of the high pressurizer pressure, and low steam line pressure safety injection, signal actuation logic at 1970 psig is required. These manual blocking features are further discussed in paragraph 7.3.1.2.2.6.
 - During further cooldown, closure and power lockout of the accumulator isolation valves is performed at 1000 psig and power lockout to the safety injection pumps is performed at approximately 220°F in the RCS.
- F. Periodic ECCS pump performance testing requires the testing of the pumps during normal power operation or at hot shutdown conditions. This precludes any potential for developing a cold overpressurization transient.
 - Should cold shutdown testing of the pumps be required, the test is done when the vessel is open to the atmosphere, again precluding overpressurization potential.
 - If cold shutdown testing with the vessel closed is necessary, the procedures require only one pump to be tested with ECCS discharge valve closure and RHRS alignment to both isolate potential ECCS pump input and to provide backup benefit of the RHRS relief valves.
 - The SI signal circuitry testing, if performed during cold shutdown, also requires RHRS alignment and safety injection pumps power lockout to preclude developing cold overpressurization transients.
- G. A steam bubble will be maintained in the pressurizer when the RCS temperature is greater than 220°F.

5.2.2.11 Consequences of a Postulated Loss of a dc Bus Coupled with a Single Failure Disabling a PORV Allowing a Cold Pressurization Event

This discussion addresses the following postulated event:

With the plant in a cooled down and depressurized condition in which the cold overpressure protection system is required to be operable and with charging and letdown established, a dc vital bus fails. This failure causes normal letdown to isolate and also results in the loss of one of the two PORVs.

In addition to the dc bus failure, an additional random failure of the second PORV is postulated to occur. This sequence of events places the plant in a condition in which letdown is isolated, the automatic cold overpressure protection system is inoperable, and charging flow is filling the pressurizer, increasing system pressure towards the Appendix G limits.

To begin this discussion the limitations placed on plant operation by the VEGP Technical Specifications are addressed.

- A. With RCS temperature below 200°F (i.e., cold shutdown) one RHR pump is required to be in operation. This requirement ensures that at least one RHR suction relief valve is available for overpressure protection of the RCS. This valve will relieve the flow from two ECCS centrifugal charging pumps and one normal centrifugal charging pump at the valve lift setting pressure.
- B. Whenever the cold overpressure protection system is required to be operable, only the two ECCS centrifugal charging pumps and the normal centrifugal charging pumps are allowed to be operable. The safety injection pumps will be inoperable. This ensures that the maximum charging letdown mismatch will be that stated in 5.2.2.10.1.
- C. Considering these requirements, anytime RHR is in operation and the RCS is in a condition requiring the cold overpressure protection to be operable, there is no overpressure event as a result of the prescribed event. Assuming the event as described^(a) did occur, the RHR relief valve would prevent RCS pressure from reaching the Appendix G limit by relieving all charging flow.

Typically at least one RHR train is in operation, or at a minimum one RHR loop suction valve is open, providing an open path from the RCS to a RHR suction relief valve, whenever RCS temperature is below 350°F. For this reason an overpressure event resulting from the prescribed event is very unlikely; however, the discussion is extended to the infrequent case where the RHRS is isolated from the RCS, and the cold overpressure protection system is required to be operable.

To gain a better understanding of the results of the event, it is necessary to address the functions of some of the chemical and volume control system control valves. As stated earlier, the letdown valves fail closed on loss of dc power isolating letdown. Between the charging pumps and the normal charging isolation valves are two normally throttled valves which receive their power from the process and control racks powered by the essential ac instrument buses. These valves are unaffected by a dc bus failure and continue to work normally during the event.

One of these valves is the charging flow control valve (1-FCV-121), which automatically regulates flow to maintain a prescribed pressurizer level. Assuming this valve continues to function normally, as pressurizer level rises, charging flow is reduced until the charging flow is limited to that required for seal injection (32 gal/min) plus a minimal amount (15 gal/min) required for regenerative heat exchanger cooling. At this flowrate, ample time is provided (as discussed below) to allow appropriate operator action. If valve control is in manual, the valve position remains unchanged. The other valve is the charging flow backpressure regulator

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⁽a) As additional information, during RHR operation letdown is typically taken from the discharge of the RHR pumps and is not isolated by the dc bus failure.

(1-HCV-182), which is manually positioned to regulate flow to the seal. This valve remains in its initial position. The effect of these two valves is to limit charging flow to its value at the beginning of the event. Assuming maximum letdown at the initiation of the event, total flow (charging plus seal injection) to the RCS is limited to approximately 130 gal/min.

An additional consideration is that, with the plant in the hot shutdown condition and RHR isolated from the RCS, normal operation is to have a steam bubble in the pressurizer of approximately 1350 ft³. At a maximum charging rate of 130 gal/min, it would take in excess of 30 min to reach the Appendix G limit at 200°F, the temperature corresponding to the coldest RCS temperature at which RHR is permitted to be isolated. As an extreme case, with a bubble of only half the normal size, the corresponding time available for appropriate action would be in excess of 15 min.

To summarize:

- A. The postulated event is unlikely to occur since the dc buses have a battery as an emergency power supply, and should the dc bus fail, it must be coupled with the additional failure of the second PORV for overpressurization.
- B. In the unlikely event that the prescribed event did occur, RHR would normally be online and capable of mitigating any potential overpressure resulting from two ECCS centrifugal charging pumps and one normal centrifugal charging pump.
- C. In the highly unlikely event that the prescribed event should occur when RHR is isolated from the RCS, the operator would have sufficient time to mitigate the event.
- D. The Appendix G curves are excessively conservative for their intended purpose of ensuring vessel integrity during cold shutdown.

No further action is necessary to address this postulated event, and the existing plant design and operational techniques result in successful event mitigation.

5.2.2.12 Testing and Inspection

Testing and inspection of the overpressure protection components are discussed in paragraph 5.4.13.4 and chapter 14. Testing capabilities of the overpressurization protection system are consistent with testing principles for systems' electronics in paragraph 7.2.2.2.3. Operational surveillance procedures will demonstrate the operability of the overpressure protection system when system operation is required. Inservice inspection is performed in accordance with Section XI of the ASME Code. Relief requests, augmented examinations, and alternate techniques will be utilized as appropriate.

5.2.2.13 References

- Cooper, L., Miselis, V., and Starek, R. M., "Overpressure Protection for Westinghouse Pressurized Water Reactors," <u>WCAP-7769</u>, Revision 1, June 1972 (also letter NS-CE-622 dated April 16, 1975, C. Eicheldinger (Westinghouse) to D. B. Vassallo (NRC), Additional Information on <u>WCAP-7769</u>, Revision 1).
- 2. Burnett, T. W. T., et al., "LOFTRAN Code Description," WCAP-7907, October 1972.

5.2.3 REACTOR COOLANT PRESSURE BOUNDARY (RCPB) MATERIALS

5.2.3.1 Material Specifications

Typical material specifications used for the principal pressure-retaining applications in Class 1 primary components and for Class 1 and 2 auxiliary components in systems required for reactor shutdown and for emergency core cooling are listed in table 5.2.3-1. Typical material specifications used for the reactor internals required for emergency core cooling, for any mode of normal operation or under postulated accident conditions, and for core structural load bearing members are listed in table 5.2.3-2.

Tables 5.2.3-1 and 5.2.3-2 may not be totally inclusive of the material specifications used in the listed applications; however, the listed specifications are representative. Identification of actual materials is available in VEGP quality assurance records.

The materials utilized conform to the applicable American Society of Mechanical Engineers (ASME) code rules.

The welding materials used for joining the ferritic base materials of the RCPB conform to or are equivalent to ASME Material Specifications SFA 5.1, 5.2, 5.5, 5.17, 5.18, and 5.20. They are qualified to the requirements of the ASME Code, Section III.

The welding materials used for joining the austenitic stainless steel base materials of the RCPB conform to ASME Material Specifications SFA 5.4 and 5.9. They are qualified to the requirements of the ASME Code, Section III.

The welding materials used for joining nickel-chromium-iron alloy in similar base material combination and in dissimilar ferritic or austenitic base material combination conform to ASME Material Specifications SFA 5.11 and 5.14. They are qualified to the requirements of the ASME Code, Section III.

5.2.3.2 Compatibility with Reactor Coolant

5.2.3.2.1 Chemistry of Reactor Coolant^a

The reactor coolant system (RCS) chemistry specifications are given in table 5.2.3-3.

The RCS water chemistry is selected to minimize corrosion. Routinely scheduled analyses of the coolant chemical composition are performed to verify that the reactor coolant chemistry meets the specifications.

The chemical and volume control system (CVCS) provides a means for adding chemicals to the RCS which perform the following functions:

- Control the pH of the coolant during prestartup testing and subsequent operation.
- Scavenge oxygen from the coolant during heatup.
- Control radiolysis reactions involving hydrogen, oxygen, and nitrogen during all power operations subsequent to startup.

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^a The Water Chemistry Control Program is credited as a license renewal aging management program (see subsection 19.2.28).

The normal limits for chemical additives and reactor coolant impurities for power operation are shown in table 5.2.3-3.

The pH control chemical utilized is lithium hydroxide monohydrate, enriched in the lithium-7 isotope to 99.9 percent. This chemical is chosen for its compatibility with the materials and water chemistry of borated water/stainless steel/zirconium/ Inconel systems. In addition, lithium-7 is produced in solution from the neutron irradiation of the dissolved boron in the coolant. The lithium-7 hydroxide is introduced into the RCS via the charging flow. The solution is prepared in the laboratory and transferred to the chemical additive tank. Reactor makeup water is then used to flush the solution to the suction header of the charging pumps. The concentration of lithium-7 hydroxide in the RCS is maintained in the range specified for pH control. If the concentration exceeds this range, the cation bed demineralizer is employed in the letdown line in series operation with the mixed bed demineralizer.

During reactor startup from the cold condition, hydrazine is employed as an oxygen scavenging agent. The hydrazine solution is introduced into the RCS in the same manner as described above for the pH control agent.

The reactor coolant is treated with dissolved hydrogen to control the net decomposition of water by radiolysis in the core region. The hydrogen also reacts with oxygen and nitrogen introduced into the RCS as impurities under the impetus of core radiation. Sufficient partial pressure of hydrogen is maintained in the volume control tank so that the specified equilibrium concentration of hydrogen is maintained in the reactor coolant. A self-contained pressure control valve maintains a minimum pressure in the vapor space of the volume control tank.

This can be adjusted to provide the correct equilibrium hydrogen concentration.

Boron, in the chemical form of boric acid, is added to the RCS for long-term reactivity control of the core.

A soluble zinc compound may be added to the reactor coolant as a means to reduce radiation fields within the primary system. The zinc used may be either natural zinc or zinc depleted of ⁶⁴Zn. When used, the target system zinc concentration is normally maintained to a concentration no greater than 40 ppb.

Suspended solid (corrosion product particulates) and other impurity concentrations are maintained below specified limits by controlling the chemical quality of makeup water and chemical additives and by purification of the reactor coolant through the CVCS.

5.2.3.2.2 Compatibility of Construction Materials with Reactor Coolant

All of the ferritic low-alloy and carbon steels which are used in principal pressure-retaining applications have corrosion-resistant cladding on all surfaces that are exposed to the reactor coolant. The corrosion resistance of the cladding material is at least equivalent to the corrosion resistance of types 304 and 316 austenitic stainless steel alloys or nickel-chromium-iron alloy, martensitic stainless steel, and precipitation-hardened stainless steel. These corrosion-resistant cladding materials may be subjected to the ASME code required postweld heat treatment for ferritic base materials.

Ferritic low-alloy and carbon steel nozzles have safe ends of either stainless steel wrought materials, stainless steel weld metal analysis A-7 (designated A-8 in the 1974 edition of the ASME code), or nickel-chromium-iron alloy weld metal F-Number 43. The latter buttering material requires further safe ending with austenitic stainless steel base material after completion of the postweld heat treatment when the nozzle is larger than a 4-in. nominal inside diameter and/or the wall thickness is greater than 0.531 in.

All of the austenitic stainless steel and nickel-chromium-iron alloy base materials with primary pressure-retaining applications are used in the solution annealed condition. These heat treatments are as required by the material specifications.

During subsequent fabrications, these materials are not heated above 800°F other than locally by welding operations. The solution-annealed surge line material is subsequently formed by hot bending followed by a resolution annealing heat treatment.

Components employing stainless steel sensitized in the manner expected during component fabrication and installation operate satisfactorily under normal plant chemistry conditions in pressurized water reactor (PWR) systems because chlorides, fluorides, and oxygen are controlled to very low levels.

5.2.3.2.3 Compatibility with External Insulation and Environmental Atmosphere

In general, all of the materials listed in table 5.2.3-1 which are used in principal pressureretaining applications and are subject to elevated temperature during system operation are in contact with thermal insulation that covers their outer surfaces.

The thermal insulation used on the RCPB is either reflective stainless steel type or made of compounded materials which yield low leachable chloride and/or fluoride concentrations. The compounded materials in the form of blocks, boards, cloths, tapes, adhesives, cements, etc., are silicated to provide protection of austenitic stainless steels against stress corrosion which may result from accidental wetting of the insulation by spillage, minor leakage, or other contamination from the environmental atmosphere. Section 1.9 indicates the degree of conformance with Regulatory Guide 1.36, Nonmetallic Thermal Insulation for Austenitic Stainless Steel.

In the event of coolant leakage, the ferritic materials will show increased general corrosion rates. Where minor leakage is anticipated from service experience, such as valve packing, pump seals, etc., only materials that are compatible with the coolant are used. These are as shown in table 5.2.3-1. Ferritic materials exposed to coolant leakage can be readily observed as part of the inservice visual and/or nondestructive inspection program to ensure the integrity of the component for subsequent service.

5.2.3.3 <u>Fabrication and Processing of Ferritic Materials</u>

5.2.3.3.1 Fracture Toughness

The fracture toughness properties of the RCPB components meet the requirements of the ASME Code, Section III, Paragraphs NB, NC, and ND-2300, as appropriate.

The fracture toughness properties of the reactor vessel materials are discussed in section 5.3.

Limiting steam generator and pressurizer reference temperature for a nil ductility transition (RT_{NDT}) temperatures are guaranteed at 60°F for the base materials and the weldments.

These materials meet the 50-ft-lb absorbed energy and 35-mils lateral expansion requirements of the ASME Code, Section III, at 120°F. The actual results of these tests are provided in the ASME material data reports which are supplied for each component and submitted to the owner at the time of shipment of the component.

Calibration of temperature instruments and Charpy impact test machines are performed to meet the requirements of the ASME Code, Section III, Paragraph NB-2360.

Westinghouse has conducted a test program to determine the fracture toughness of low-alloy ferritic materials with specified minimum yield strengths greater than 50,000 psi to demonstrate compliance with Appendix G of the ASME Code, Section III. In this program, fracture toughness properties were determined and shown to be adequate for base metal plates and forgings, weld metal, and heat-affected zone metal for higher strength ferritic materials used for components of the RCPB. The results of the program are documented in reference 1, which has been submitted to the Nuclear Regulatory Commission (NRC) for review.

5.2.3.3.2 Control of Welding

All welding is conducted utilizing procedures qualified according to the rules of Sections III and IX of the ASME Code. Control of welding variables, as well as examination and testing, during procedure qualification and production welding is performed in accordance with ASME Code requirements.

Westinghouse practices for storage and handling of welding electrodes and fluxes comply with ASME Code, Section III, Paragraph NB-2400.

Section 1.9 indicates the degree of conformance of the ferritic materials components of the RCPB with Regulatory Guides 1.34, Control of Electroslag Weld Properties, 1.43, Control of Stainless Steel Weld Cladding of Low-Alloy Steel Components, 1.50, Control of Preheat Temperature for Welding of Low-Alloy Steel, and 1.71, Welder Qualification for Areas of Limited Accessibility.

5.2.3.4 Fabrication and Processing of Austenitic Stainless Steel

Paragraphs 5.2.3.4.1 through 5.2.3.4.5 address Regulatory Guide 1.44, Control of the Use of Sensitized Stainless Steel, and present the methods and controls utilized by Westinghouse to avoid sensitization and prevent intergranular attack (IGA) of austenitic stainless steel components. Also, section 1.9 indicates the degree of conformance with Regulatory Guide 1.44.

5.2.3.4.1 Cleaning and Contamination Protection Procedures

It is required that all austenitic stainless steel materials used in the fabrication, installation, and testing of nuclear steam supply components and systems be handled, protected, stored, and cleaned according to recognized and accepted methods which are designed to minimize contamination which could lead to stress corrosion cracking. The rules covering these controls are stipulated in Westinghouse process specifications. As applicable, these process specifications supplement the equipment specifications and purchase order requirements of every individual austenitic stainless steel component or system which Westinghouse procures for the VEGP nuclear steam supply systems (NSSSs), regardless of the ASME Code classification. Westinghouse process specifications are also given to Bechtel, the architectengineer, and to Georgia Power Company for recommended use within their scope of supply and activity.

The process specifications that define these requirements and that follow the guidance of the American National Standards Institute (ANSI) N-45 committee specifications include the following:

<u>Number</u>	Process Specification
82560HM	Requirements for Pressure Sensitive Tapes for Use on Austenitic Stainless Steels.
83336KA	Requirements for Thermal Insulation Used on Austenitic Stainless Steel Piping and Equipment.
83860LA	Requirements for Marking of Reactor Plant Components and Piping.
8435OHA	Site Receiving Inspection and Storage Requirements for Systems, Material, and Equipment.
8435INL	Determination of Surface Chloride and Fluoride on Austenitic Stainless Steel Materials.
853I0QA	Packaging and Preparing Nuclear Components for Shipment and Storage.
292722	Cleaning and Packaging Requirements of Equipment for Use in the NSSS.
597756	Pressurized Water Reactor Auxiliary Tanks Cleaning Procedures.
597760	Cleanliness Requirements During Storage Construction, Erection, and Startup Activities of Nuclear Power System.

Section 1.9 indicates the degree of conformance of the austenitic stainless steel components of the RCPB with Regulatory Guide 1.37, Quality Assurance Requirements for Cleaning of Fluid Systems and Associated Components of Water-Cooled Nuclear Power Plants.

5.2.3.4.2 **Solution Heat Treatment Requirements**

The austenitic stainless steels listed in tables 5.2.3-1 and 5.2.3-2 are utilized in the final heattreated condition required by the respective ASME Code, Section II materials specification for the particular type or grade of alloy.

5.2.3.4.3 **Material Testing Program**

Westinghouse practice is that austenitic stainless steel materials of product forms with simple shapes need not be corrosion tested provided that the solution heat treatment is followed by water quenching. Simple shapes are defined as all plates, sheets, bars, pipe, and tubes, as well as forgings, fittings, and other shaped products that do not have inaccessible cavities or chambers that would preclude rapid cooling when water quenched. When testing is required, the tests are performed in accordance with ASTM A 262, Practice A or E, as amended by Westinghouse Process Specification 84201MW.

5.2.3.4.4 Prevention of Intergranular Attack of Unstabilized Austenitic Stainless Steels

Unstabilized austenitic stainless steels are subject to IGA provided that three conditions are present simultaneously. These are:

- An aggressive environment; e.g., an acidic aqueous medium containing chlorides or oxygen.
- A sensitized steel.
- A high temperature.

If any one of the three conditions described above is not present, IGA will not occur. Since high temperatures cannot be avoided in all components in the NSSS, reliance is placed on the elimination of the other two conditions to prevent IGA on wrought stainless steel components.

This is accomplished by:

- Control of primary water chemistry to ensure a benign environment.
- Utilization of materials in the final heat-treated condition and the prohibition of subsequent heat treatments in the 800°F and 1500°F temperature range.
- Control of welding processes and procedures to avoid heat-affected zone sensitization.
- Confirmation that the welding procedures used for the manufacture of components in the primary pressure boundary and the reactor internals do not result in the sensitization of heat-affected zones.

Further information on each of these steps is provided in the following paragraphs.

The water chemistry in the RCS is controlled to prevent the intrusion of aggressive species. In particular, the maximum permissible oxygen and chloride concentrations are 0.1 ppm and 0.15 ppm, respectively. Table 5.2.3-3 lists the recommended reactor coolant water chemistry specifications. The precautions taken to prevent the intrusion of chlorides into the system during fabrication, shipping, and storage are stipulated in the appropriate process specifications. The use of hydrogen overpressure precludes the presence of oxygen during operation. The effectiveness of these controls has been demonstrated by both laboratory tests and operating experience. The long-term exposure of severely sensitized stainless steels to reactor coolant environments in early Westinghouse PWRs has not resulted in any sign of IGA. Reference 2 describes the laboratory experimental findings and reactor operating experience. The additional years of operations since the issuing of reference 2 have provided further confirmation of the earlier conclusions that severely sensitized stainless steels do not undergo any IGA in Westinghouse PWR coolant environments.

Although there is no evidence that PWR coolant water attacks sensitized stainless steels, Westinghouse considers it good metallurgical practice to avoid the use of sensitized stainless steels in the NSSS components. Accordingly, measures are taken to prohibit the purchase of sensitized stainless steels and to prevent sensitization during component fabrication. Wrought austenitic stainless steel stock is used for components that are part of:

- The RCPB.
- Systems required for reactor shutdown.
- Systems required for emergency core cooling.

 Reactor vessel internals relied upon to permit adequate core cooling for normal operation or under postulated accident conditions.

The wrought austenitic stainless steel stock is utilized in one of the following conditions:

- Solution annealed and water quenched.
- Solution annealed and cooled through the sensitization temperature range within less than approximately 5 min.

It is generally accepted that these practices will prevent sensitization. Westinghouse has verified this by performing corrosion tests on wrought material as it was received.

The heat-affected zones of welded components must, of necessity, be heated into the sensitization temperature range, 800°F to 1500°F. However, severe sensitization (i.e., continuous grain boundary precipitates of chromium carbide, with adjacent chromium depletion) can be avoided by controlling welding parameters and welding processes. The heat input and associated cooling rate through the carbide precipitation range are of primary importance. Westinghouse has demonstrated this by corrosion testing a number of weldments.

Heat input is calculated according to the formula:

$$H = \frac{(E)(I)(60)}{S}$$

where:

H = joules/in.

E = volts.

I = amperes.

S = travel speed (in./min).

Of 25 production and qualification weldments tested, representing all major welding processes and a variety of components and incorporating base metal thicknesses from 0.10 to 4.0 in., only portions of two were severely sensitized. Of these, one involved a heat input of 120,000 J, and the other involved a heavy socket weld in relatively thin-walled material. In both cases, sensitization was caused primarily by high-heat inputs relative to the section thickness. In only the socket weld did the sensitized condition exist at the surface, where the material is exposed to the environment. The component has been redesigned, and a material change has been made to eliminate this condition.

The heat input in all austenitic pressure boundary weldments has been controlled by:

- Prohibiting the use of block welding.
- Limiting the maximum interpass temperature to 350°F.
- Exercising approval rights on all welding procedures.

5.2.3.4.5 Retesting Unstabilized Austenitic Stainless Steels Exposed to Sensitization Temperatures

As described in the previous section, it is not normal Westinghouse practice to expose unstabilized austenitic stainless steels to the sensitization range of 800°F to 1500°F during fabrication into components. If during the course of fabrication, the steel is inadvertently exposed to the sensitization temperature range, 800°F to 1500°F, the material may be tested in

accordance with ASTM A 262, as amended by Westinghouse Process Specification 84201MW, to verify that it is not susceptible to IGA, except that testing is not required for:

- A. Cast metal or weld metal with a ferrite content of 5 percent or more.
- B. Material with a carbon content of 0.03 percent or less that is subjected to temperatures in the range of 800°F to 1500°F for less than 1 h.
- C. Material exposed to special processing, provided the processing is properly controlled to develop a uniform product and provided that adequate documentation exists of service experience and/or test data to demonstrate that the processing will not result in increased susceptibility to intergranular stress corrosion.

If it is not verified that such material is not susceptible to IGA, the material is resolution annealed and water quenched or rejected.

5.2.3.4.6 Control of Welding

The following paragraphs address Regulatory Guide 1.31, Control of Ferrite Content in Stainless Steel Weld Metal, and present the methods used, and the verification of these methods, for austenitic stainless steel welding.

The welding of austenitic stainless steel is controlled to mitigate the occurrence of microfissuring or hot cracking in the weld. Although published data and experience have not confirmed that fissuring is detrimental to the quality of the weld, it is recognized that such fissuring is undesirable in a general sense. Also, it has been well documented in the technical literature that the presence of delta ferrite is one of the mechanisms for reducing the susceptibility of stainless steel welds to hot cracking. However, there is insufficient data to specify a minimum delta ferrite level below which the material will be prone to hot cracking. It is assumed that such a minimum lies somewhere between 0- and 3-percent delta ferrite.

The scope of these controls discussed herein encompasses welding processes used to join stainless steel parts in components designed, fabricated, or stamped in accordance with the ASME Code, Section III, Class 1 and 2, and core support components. Delta ferrite control is appropriate for the above welding requirements, except where no filler metal is used or where for other reasons such control is not applicable. These exceptions include electron beam welding, autogenous gas shielded tungsten arc welding, explosive welding, and welding using fully austenitic welding materials.

The fabrication and installation specifications require welding procedure and welder qualification in accordance with Section III and include the delta ferrite determinations for the austenitic stainless steel welding materials that are used for welding qualification testing and for production processing. Specifically, the undiluted weld deposits of the "starting" welding materials are required to contain a minimum of 5-percent delta ferrite (the equivalent ferrite number may be substituted for percent delta ferrite) as determined by chemical analysis and calculation using the appropriate weld metal constitution diagrams in Section III. When new welding procedure qualification tests are evaluated for these applications, including repair welding of raw materials, they are performed in accordance with the requirements of Sections III and IX.

The results of all the destructive and nondestructive tests are reported in the procedure qualification record in addition to the information required by Section III.

The starting welding materials used for fabrication and installation welds of austenitic stainless steel materials and components meet the requirements of Section III. The austenitic stainless steel welding material conforms to ASME weld metal analysis A-7 (designated A-8 in the 1974 edition of the ASME Code), type 308 or 308L for all applications. Bare weld filler metal, including consumable inserts, used in inert gas welding processes conform to ASME SFA 5.9, and are procured to contain not less than 5-percent delta ferrite according to Section III. Weld filler metal materials used in flux shielded welding processes conform to ASME SFA 5.4 or 5.9 and are procured in a wire-flux combination to be capable of providing not less than 5-percent delta ferrite in the deposit according to Section III. Welding materials are tested using the welding energy inputs to be employed in production welding.

Combinations of approved heat and lots of starting welding materials are used for all welding processes. The welding quality assurance program includes identification and control of welding material by lots and heats as appropriate. All of the weld processing is monitored according to approved inspection programs which include review of starting materials, qualification records, and welding parameters. Welding systems are also subject to:

- Quality assurance audit including calibration of gauges and instruments.
- Identification of starting and completed materials.
- Welder and procedure qualifications.
- Availability and use of approved welding and heat-treating procedures.
- Documentary evidence of compliance with materials, welding parameters, and inspection requirements.

Fabrication and installation welds are inspected using nondestructive examination methods according to Section III rules.

To ensure the reliability of these controls, Westinghouse has completed a delta ferrite verification program, described in reference 3. This program has been approved as a valid approach to verify the Westinghouse hypothesis and is considered an acceptable alternative for conformance with the NRC Interim Position on Regulatory Guide 1.31. The Regulatory Staff's acceptance letter and topical report evaluation were received on December 30, 1974. The program results, which do support the hypothesis presented in reference 3, are summarized in reference 4.

Section 1.9 indicates the degree of conformance of the austenitic stainless steel components of the RCPB with Regulatory Guides 1.34, Control of Electroslag Properties, and 1.71, Welder Qualification for Areas of Limited Accessibility.

5.2.3.5 References

- 1. Logsdon, W. A., Begley, J. A., and Gottshall, C. L., "Dynamic Fracture Toughness of ASME SA-508 Class 2a and ASME SA-533 Grade A Class 2 Base and Heat-Affected Zone Material and Applicable Weld Metals," <u>WCAP-9292</u>, March 1978.
- 2. Golik, M. A., "Sensitized Stainless Steel in Westinghouse PWR Nuclear Steam Supply Systems," <u>WCAP-7477-L</u> (Proprietary), March 1970, and <u>WCAP-7735</u> (Nonproprietary), August 1971.
- 3. Enrietto, J. F., "Control of Delta Ferrite in Austenitic Stainless Steel Weldments," <u>WCAP-</u>8324-A, June 1975.

4. Enrietto, J. F., "Delta Ferrite in Production Austenitic Stainless Steel Weldments," WCAP-8693, January 1976.

5.2.4 INSERVICE INSPECTION AND TESTING OF REACTOR COOLANT PRESSURE BOUNDARY^a

Inservice inspection and testing of Class 1 pressure-retaining components such as vessels, piping, pumps, valves, bolting, and supports within the reactor coolant pressure boundary shall be performed in accordance with Section XI of the American Society of Mechanical Engineers (ASME) Code including any applicable addenda in accordance with 10 CFR 50.55a(g)(4)(ii) (specific edition and any applicable addenda of the code will be delineated in each program), with certain exceptions whenever specific written relief is granted by the Nuclear Regulatory Commission (NRC) in accordance with 10 CFR 50.55a(a)3 and 10 CFR 50.55a(g)(6)(i). The inservice testing of pumps and valves in accordance with the requirements of Articles IWP and IWV of the code is discussed in subsection 3.9.6. Class 2 and 3 components examinations are addressed in section 6.6.

The preservice inspection program requirements for each unit were completed prior to the commercial operation date for each of the respective units. The preservice inspection program for Unit 1 complied with the ASME Code, Section XI, 1980 Edition including addenda through Winter 1980, except that reactor pressure vessel examinations were performed using the 1980 Edition including addenda through Winter 1981. The preservice inspection program for Unit 2 complied with the ASME Code, Section XI, 1983 Edition including addenda through Summer 1983, except that reactor pressure vessel examinations were performed using the 1980 Edition including addenda through Winter 1981. Certain preservice inspection requirements of the ASME Code. Section XI were determined to be impractical and relief requests were granted by the NRC, pursuant to 10 CFR 50.55a(g)(i). The relief requests were supported by information pursuant to 10 CFR 50.55a (a) (3). The inservice inspection program and inservice test program were submitted to the NRC prior to commercial operation. These programs comply with applicable inservice inspection provisions of 10 CFR 50.55a(g) and the NRC guidelines attached as an appendix to section 121.0 of review questions entitled "Guidance for Preparing Preservice and Inservice Inspection Programs and Relief Requests Pursuant to 10 CFR 50.55a(q)." Where compliance with code requirements is not practical, relief requests have been submitted to the NRC for review and approval. The inservice programs detail the areas subject to examination and method, extent, and frequency of examinations. Additionally, component supports and snubber testing requirements are included in the inspection programs.

5.2.4.1 System Boundary Subject to Inspection

In addition to the reactor pressure vessel, all Class 1 components such as vessels, piping, pumps, valves, bolting, and supports shall be inspected to the extent practical, in accordance with Article IWB of ASME Code, Section XI. Class 1 pressure-retaining components and their specific boundaries are identified in the inspection plan documents.

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^a The Inservice Inspection Program is credited as a license renewal aging management program (see subsection 19.2.13).

5.2.4.2 <u>Arrangement and Accessibility</u>

The physical arrangement of components was designed to allow personnel and equipment access to the extent practical to perform the required inservice examinations. Removable insulation and shielding was provided on those piping systems requiring volumetric and surface examination. Temporary or permanent working platforms, scaffolding, and ladders are provided to facilitate access to piping welds.

An inservice inspection design review was undertaken to identify exceptions to the access requirements of the code with subsequent design modifications and/or inspection technique development to ensure code compliance to the extent practical. Additional exceptions may be identified and reported to the NRC after plant operation, as specified in 10 CFR 50.55a(g)(5)(iv).

Space has been provided to handle and store insulation, structural members, shielding, and other materials related to the inspection. Suitable hoists and other handling equipment, lighting, and sources of power for inspection equipment were installed at appropriate locations. The reactor pressure vessel (RPV) inspections are performed primarily from the vessel internal surfaces. Other areas of the RPV such as the closure head are accessible from the outer surfaces of the vessel for inspection. Closure studs, nuts, and washers are removed to a dry location for direct inspection.

5.2.4.3 Examination Techniques and Procedures

The visual, surface, and volumetric examination techniques, procedures, and special techniques are in accordance with the requirements of subarticle IWA-2200 and table IWB-2500-1 of the ASME Code, Section XI except where compliance with code requirements is not practical and relief has been requested from the NRC. Liquid penetrant methods and/or magnetic particle methods are used for surface examinations. Radiography and/or ultrasonic techniques, whether manual or remote, are used for volumetric examinations. A special vessel inspection tool is used to inspect the RPV welds from the vessel internal surfaces. Welds located in the reactor vessel beltline region are examined to meet the requirements of Regulatory Guide 1.150 to the extent practicable. Other RPV welds are examined to meet the requirements of Regulatory Guide 1.150, with the exception of the near surface examination, to the extent practicable. Other examination techniques may be used provided that the results are demonstrated to be equivalent or superior to the above techniques.

5.2.4.4 <u>Inspection Intervals</u>

Inspection intervals are as defined in subarticles IWA-2400 and IWB-2400 of ASME Code, Section XI. The interval may be extended by as much as 1 year to permit inspections to be concurrent with plant outages. It is intended that inservice examinations be performed during normal plant outages such as refueling shutdowns or maintenance shutdowns occurring during the inspection interval.

5.2.4.5 Examination Categories and Requirements

The examination categories and requirements are in accordance with subarticle IWB-2500 and table IWB-2500-1 of ASME Code, Section XI. The preservice examinations complied with IWB-2200.

5.2.4.6 **Evaluation of Examination Results**

Examination results are evaluated in accordance with IWB-3000, with flaw indications in accordance with IWB-3400 and table IWB-3410. Repair procedures are in accordance with IWB-4000 of ASME Code, Section XI.

5.2.4.7 System Leakage and Hydrostatic Pressure Tests

System pressure tests comply with IWA-5000 and IWB-5000 of ASME Code, Section XI.

5.2.5 DETECTION OF LEAKAGE THROUGH REACTOR COOLANT PRESSURE BOUNDARY

The reactor coolant pressure boundary (RCPB) leakage detection systems monitor leaks from the reactor coolant and associated systems. These systems provide information which permit the plant operators to take corrective action if a leak is evaluated as detrimental to the safety of the facility.

5.2.5.1 Design Bases

The leak detection systems are designed in accordance with the requirements of 10 CFR 50 and the general design criterion 30 to provide a means of detecting and, to the extent practical, identifying the source of the reactor coolant leakage. The systems conform with Regulatory Guide 1.45. Main systems that monitor the environmental condition of the containment include the sump level monitoring system, the airborne particulate radioactivity monitoring systems, and the containment fan cooler condensate measuring system. In addition to the above systems, the humidity, temperature, pressure, and radiogas monitors provide indirect indication of leakage to the containment.

Associated systems and components connected to the reactor coolant system have intersystem leakage monitoring devices.

These leakage detection systems are qualified for all seismic events not requiring a shutdown. The airborne radioactivity monitoring system is qualified for a safe shutdown earthquake (SSE).

5.2.5.1.1 Leakage Classification

RCPB leakage is classified as either identified or unidentified leakage. Identified leakage includes: leakage into closed systems, such as pump seal or valve packing leaks that are captured, flow metered, and conducted to a sump or collecting tank; or leakage into the containment atmosphere from sources that are both specifically located and known either not to interfere with the operation of unidentified leakage monitoring systems or not to be from a flaw in the RCPB; or leakage into auxiliary systems and secondary systems. Unidentified leakage is all other leakage.

5.2.5.1.2 Limits for Reactor Coolant Leakage

Limits for reactor coolant leakage are identified in the Technical Specifications.

5.2.5.2 <u>Identified Intersystem Leakage Detection</u>

Unidentified leakage into closed primary systems is directed to the reactor coolant drain tank or pressurizer relief tank. Identified leakage, such as pump seal or valve packing leakage, is directed to the reactor coolant drain tank where it is monitored by tank pressure, temperature, level, and flow instrumentation on the reactor coolant drain tank discharge lines.

Identified leakage, such as leakage past the pressurizer safety valves or power-operated relief valves (PORVs), is directed to the pressurizer relief tank. This leakage is monitored by temperature instrumentation in the piping system and tank pressure, temperature, and level instrumentation. Leakage collected in the pressurizer relief tank is directed to the reactor coolant drain tank for subsequent treatment and discharge.

An important identified leakage path for reactor coolant into other systems is through the steam generator tubes into the secondary side of the steam generator. Identified leakage to the steam generators is detected by means of the steam generator sample liquid or condenser air ejector radiation monitors. Two additional primary-to-secondary leak detection systems are also provided: a noble gas detector and a system utilizing N16 as the detection medium. The N16 detector is installed in the turbine building main steam pipe chase, between the two main steam pipes. The N16 leak monitor is independent of the primary loop fission and corrosion product radioactivity, so when the tube leak rates increase without considerable changes in the secondary side radioactivity levels, the system can still detect small leaks. The noble gas detector is located in the condenser steam jet air ejector discharge header immediately prior to the filtration unit. For details of these radiation monitors, see subsection 11.5.2.

Auxiliary systems connected to the RCPB incorporate design and administrative provisions that serve to limit leakage. These provisions include isolation valves designed for low seat leakage, periodic testing of RCPB check valves (paragraph 6.3.4.2), and inservice inspection (subsection 5.2.4 and section 6.6). Leakage is detected by increasing auxiliary system level, temperature, and pressure indications or lifting of relief valves accompanied by increasing values of monitored parameters in the relief valve discharge path. These systems are isolated from the RCS by normally closed valves and/or check valves.

5.2.5.2.1 Description and Operation of Identified Leak Detection System

A. Residual Heat Removal System (RHRS) (Suction Side)

The RHRS is isolated from the RCS on the suction side by motor-operated valves HV-8701A/B and HV-8702A/B. Leakage past these valves is detected by lifting of relief valves PSV-8708A or PSV-8708B, accompanied by increasing pressurizer relief tank level, pressure, and temperature indications and alarms on the main control board.

B. Safety Injection System (SIS)/Accumulators

The accumulators are isolated from the RCS by check valves 1204-U6-083 through -086 and 1204-U6-079 through -082. Leakage past these valves and into the accumulator subsystem is detected by redundant control room accumulator pressure and level indications and alarms.

C. SIS/RHR Discharge Subsystem

The RHR pump portion of the SIS is isolated from the RCS by check valves 1204-U6-083 through -086, 1204-U6-147 through -150, 1204-U6-125 and -126, 1204-U6-128 and -129, and normally closed motor-operated valve HV-8840.

Leakage past these valves will eventually pressurize the RHR discharge header and the pump suction header through the normally open pump miniflow isolation valves FV-610 or FV-611. A continued increase in RHR pump discharge pressure will be indicated in the control room and ultimately result in lifting relief valves PSV-8708A and PSV-8708B in the suction header.

D. SIS/Safety Injection Pump Subsystem

The safety injection pump portion of the SIS is isolated from the RCS by check valves 1204-U4-083 through -086, 1204-U4-143 through -146, 1204-U6-124 through -127, 1204-U4-120 through -123, and normally closed motor- operated valves HV-8802A/B. Leakage past these valves will pressurize the safety injection pump discharge header, resulting in control room indication of increasing pressure and, eventually, lifting of relief valve PSV-8851 or PSV-8853A/B.

E. SIS/Centrifugal Charging Pump Subsystem

The charging pump subsystem of the SIS is isolated from the RCS by check valves 1204-U4-026 through -029, 1204-U6-013, and motor-operated valves HV-8801A/B. Leakage past valves HV-8801A/B is not possible, since the valve inlets are pressurized by the operating charging pump.

F. Head Gasket Monitoring Connections

The reactor vessel flange and head are sealed by two metallic O-rings. These gaskets are of the hollow self-energizing type in which pressure of the fluid being sealed enters the interior of the gasket. The O-rings are fastened to the closure head by a mechanical connection to facilitate removal.

Seal leakage is detected by means of two leak-off connections: one between the inner and outer ring, and one outside the outer O-ring. A manual isolation valve is installed just outside the missile barrier of each leak-off line. Downstream of these valves the lines are headered before being routed to the reactor coolant drain tank in the waste processing system. An air-operated isolation valve, actuated from the control board, is installed in the common line. During normal plant operation, the leak-off piping is aligned such that leakage across the inner O-ring passes through valves 1201-U4-088 and HV-8032 into the drain tank. A surface mounted, resistance temperature detector, installed on the bottom of the common pipe, signals leakage at an alarm setpoint. A blind flanged branch line containing isolation valve 1201-U4-089 is provided to confirm and establish the magnitude of the leakage.

Once inner O-ring leakage is discovered, valve 1201-U4-087 should be opened and valve 1201-U4-088 closed so that possible leakage across the second O-ring would be monitored.

In addition, during plant refueling operations both the inner and outer reactor vessel flange leak-off valves are closed. This prevents possible gas leakage from the reactor coolant drain tank to the containment atmosphere. Refer to drawings 1X4DB111 and 2X4DB111 for the flow diagram representation.

The reactor vessel is the only flanged vessel within the RCPB that is provided with leak-off collection provisions.

Leakage past the reactor vessel head gaskets results in temperature indication and alarm in the control room.

G. Component and Auxiliary Component Cooling Water Systems

Leakage from the RCS to the component cooling water (CCW) and auxiliary component cooling water (ACCW) systems, which service all RCPB associated components that require cooling, is detected by the CCW and ACCW radioactivity monitoring system (subsection 11.5.2) and/or increasing surge tank level. Components serviced by these auxiliary cooling systems include: reactor coolant pump thermal barriers, RHR heat exchangers, letdown line heat exchangers, reactor coolant pump seal water heat exchangers.

5.2.5.3 Unidentified Leakage Detection

Normally, unidentified leakage from the RCS is very low. The RCS is an all-welded system, with the exception of the connections on the pressurizer safety valves, reactor vessel head, pressurizer and steam generator manways, and reactor vessel head vent, which are flanged.

In general, valves in the RCS that are 2 in. and under are of the packless type. All valves larger than 2 in. have dual packing with a leak-off connection to the reactor coolant drain tank between the two packings or a reduced packing configuration with the valve stem leakoff line capped.

Primary indications of unidentified coolant leakage to the containment are provided by air particulate radioactivity monitors, gaseous radioactivity monitors, fan cooler condensate flow monitors, and containment sump level monitors.

In normal operation, these primary monitors show a background level that is indicative of the normal level of unidentified leakage inside the containment. Variations in airborne radioactivity or specific humidity above the normal level signify an increase in unidentified leakage rates and signal to the plant operators that corrective action may be required. Similarly, increases in containment sump level signify an increase in unidentified leakage.

RCS unidentified leakage may also be indicated by increasing charging pump flowrate compared with normal RCS inventory changes and by unscheduled increases in reactor makeup water usage.

Reactor coolant inventory monitoring provides an indication of system leakage. Net level changes in the pressurizer and volume control tank are indicative of system leakage, since the chemical and volume control system is a closed loop connected to the RCS. Monitoring net makeup to the chemical and volume control system, as well as net collected leakage, provides an important method of obtaining information for use in establishing a water inventory balance. An abnormal increase in makeup water requirements or a significant change in the water inventory balance can be indicative of increased system leakage.

The sensitivity and response time of the detection equipment for unidentified leakage is such that a leakage rate, or its equivalent, of 1 gal/min can be detected in approximately 1 h.

The above methods are supplemented by visual and ultrasonic inspections of the RCPB during plant shutdown periods, in accordance with the inservice inspection program (subsection 5.2.4).

5.2.5.3.1 Description and Operation of Main Unidentified Leak Detection Systems

Systems employed for detecting leakage to the containment from unidentified sources are:

- Containment airborne particulate radioactivity monitor.
- Containment gaseous radioactivity monitor.

- Containment air cooler condensate flow monitor.
- Containment sump level monitor.

Additionally, humidity, temperature, and pressure monitoring of the containment atmosphere are used for alarms and indirect indication of leakage to the containment.

A. Containment Airborne Particulate Radioactivity Monitoring System

An air sample is drawn outside the containment into a closed system by a sample pump and is then consecutively passed through a particulate filter with detector and a gaseous monitor chamber with detector. The filter collects 99 percent of the particulate matter greater than 1 mm in size. The sample transport system includes:

- A pump to obtain the air sample.
- A flow control valve to provide flow adjustment.
- A flow meter to indicate the flowrate.
- A flow alarm assembly to provide high- and low-flow alarm signals.

The particulate filter is continuously monitored by a scintillation crystal with a photomultiplier tube that provides an output signal proportional to the activity collected on the filter. The particulate monitor has a minimum detectable concentration of $10^{-11} \, \mu \text{Ci/cm}^3$ and a range of $10^{-11} \, \text{to} \, 10^{-6} \, \mu \text{Ci/cm}^3$. More details concerning the particulate monitors can be found in subsection 11.5.2.

Particulate activity can be correlated with the coolant fission and corrosion product activities. Any increase of more than two standard deviations above the count rate for background would indicate a possible leak. The total particulate activity concentration above background, due to an abnormal leak and natural decay, increases almost linearly with time for the first several hours after the beginning of a leak. As shown in figure 5.2.5-1, with 0.01-percent failed fuel, containment background airborne particulate radioactivity equivalent to 10^{-3} percent/day, and a partition factor equal to 0.001, a 1-gal/min leak would be detected in approximately 1 h. Larger leaks would be detected in proportionately shorter times (exclusive of sample transport time, which remains constant). The detection capabilities and response times are shown in figure 5.2.5-1.

The activity is indicated on displays and electronically recorded. High-activity alarm indications are displayed on the radiation monitoring cabinets. Local alarms provide operational status of supporting equipment such as pumps, motors, and flow and pressure controllers.

The leakage flowrate can be determined by performing a water inventory balance of the reactor coolant system when the count rate indicates a possible leak as explained above.

B. Containment Gaseous Radioactivity Monitoring System

The containment gaseous radioactivity monitor determines gaseous radioactivity in the containment by monitoring continuous air samples from the containment atmosphere. After passing through the gas monitor, the sample is returned via the closed system to the containment atmosphere.

Each sample is continuously mixed in a fixed, shielded volume where its activity is monitored. The monitor has a range of 10^{-7} to 10^{-2} µCi/cm³ and a minimum detectable concentration of 5 x 10^{-7} µCi/cm³.

The containment gaseous radioactivity monitors are described in subsection 11.5.2.

Gaseous radioactivity can be correlated with the gaseous activity of the reactor coolant. Any increase more than two standard deviations above the count rate for background would indicate a possible leak. The total gaseous activity level above background increases almost linearly for the first several hours after the beginning of the leak. As specified in figure 5.2.5-1, with 0.01-percent failed fuel, containment background airborne gaseous radioactivity equivalent to 1 percent/day, and a partition factor equal to 1, a 1-gal/min leak would be detected in approximately 1 h. Larger leaks would be detected in proportionately shorter times (exclusive of the sample transport time, which remains constant). The detection capabilities and response times are shown in figure 5.2.5-1.

The detector outputs are transmitted to the radiation monitoring system cabinets in the control room, where the activity is indicated by displays and electronically recorded. High-activity alarm indications are displayed on the control board annunciator in addition to the radiation monitoring system cabinets. Local alarms annunciate the operational status of the supporting equipment.

The leakage flowrate can be determined by performing a water inventory balance of the reactor coolant system when the count rate indicates a possible leak as explained above.

The containment purge system radioactivity monitors (subsection 11.5.2) serve as backup to the containment air particulate and gaseous airborne radioactivity monitoring system while the purge is in operation.

The containment purge monitors function in the same manner as the containment air particulate and gaseous radioactivity monitors, except that the purge monitors sample from the containment purge exhaust line.

C. Containment Air Cooler Condensate Monitoring System

The condensate monitoring system permits measurements of the liquid runoff from the containment cooler units. It consists of a containment cooler drain collection header, a vertical standpipe, valving, and standpipe level instrumentation. The condensation from the containment coolers flows via the collection header to the vertical standpipe. A differential pressure transmitter provides standpipe level signals. The system provides measurements of low leakages by monitoring standpipe level increase versus time.

The condensate flowrate is a function of containment humidity, nuclear service cooling water (NSCW) temperature, and containment purge rate. The water vapor dispersed by a 1-gal/min leak is usually greater than the water vapor brought in with the outside air. Air brought in from the outside is heated to 60°F before it enters the containment.

After the air enters the containment, it mixes with the containment atmosphere and is heated to between 100°F and 120°F while the relative humidity drops. The most important factor in condensing the water vapor is the temperature of the NSCW which is supplied to the containment coolers. This water is assumed to vary in temperature between 35°F and 95°F.

Drainage flowrate from the units due to normal condensation is calculated for the ambient (background) atmospheric conditions present within the containment. With the initiation of an additional or abnormal leak, the containment atmosphere humidity and condensation runoff rate both begin to increase, the water level rises in the vertical pipe, and the high condensate flow alarm is actuated. Level changes of as little as 0.25 in. in the cooler condensate standpipes can be detected.

Figure 5.2.5-1 shows the detection capabilities of the system for various conditions. Normal background leakage increases containment humidity to the point where the condensation rate will increase, which improves the detection capabilities of this system. As shown in figure 5.2.5-1, a sensitivity of 1 gal/min in approximately 1 h can be achieved when supplying cold NSCW to the containment coolers or with the initial background leakage.

The rate of leakage can be determined when the precise NSCW, outside air, and containment air temperatures, and the outside relative humidity are known.

D. Containment Sump Level Monitoring System

Since a leak in the primary system would result in reactor coolant flowing into the containment normal or reactor cavity sumps, leakage would be indicated by a level increase in the sump. Indication of increasing sump level is transmitted from the sump to the control room level indicator by means of a sump level transmitter. The system provides measurements of low leakages by monitoring level increase versus time.

The detection capabilities of the containment normal sump and reactor cavity sump are shown in figure 5.2.5-1, assuming that the water from the leak is collected in the sump.

The actual reactor coolant leakage rate can be established from the increase above the normal rate of change of sump level. A check of other instrumentation would be required to eliminate possible leakage from nonradioactive systems as a cause of an increase in sump level. The leakage rate can also be determined from the frequency of sump pump operation.

Under normal conditions, the containment normal sump pumps operate infrequently and reactor cavity sump pump operates very infrequently. Gross leakage can be surmised from unusual frequency of pump operation. Sump level and pump running indication are provided in the control room to alert the operators.

5.2.5.3.2 Additional Unidentified Leakage Detection Methods

Other methods available for detecting leakage are:

A. Charging Pump Operation

During normal operation, one of the charging pumps is in operation. If a gross increase in reactor coolant leakage occurs, the flowrate of the charging pump would increase, indicating leakage from the RCS. This leakage must be sufficient to cause a decrease in pressurizer or volume control tank level that is within the sensitivity range of the level indicators. The flowrate of the charging

pump would automatically increase to try to maintain pressurizer level. Charging pump discharge flow indication is provided in the control room.

The leakage rate can be determined by the amount that the charging pump flowrate increases above the letdown flowrate to maintain constant pressurizer level. Any significant increase in the charging flowrate is a possible indication of a leak.

B. Containment Humidity Monitoring System

The containment humidity system, utilizing temperature-compensated humidity detectors, is provided to determine the water-vapor content of the containment atmosphere. An increase in the humidity of the containment atmosphere indicates release of water within the containment. The range of the containment humidity measuring system is 5- to 99-percent relative humidity at 80°F with a temperature range of 40 to 120°F. The accuracy of the humidity detectors is ± 3 percent.

The response of the containment humidity under various outside air conditions and no leakage falls within the extremes shown in figure 5.2.5-1. The humidity monitor supplements the condensate monitor. It is most sensitive under conditions when there is no condensation.

A rapid increase of humidity over the background level by more than 10 percent can be taken as a probable indication of a leak.

The leakrate can be determined when the outside air temperature and humidity and the containment atmosphere temperature are known.

C. Liquid Inventory

The operators can surmise gross leakage from changes in the reactor coolant inventory. Noticeable decreases in the pressurizer level not associated with known changes in operation are investigated. Likewise, makeup water usage information which is available from the plant computer is checked frequently for unusual makeup rates not due to plant operations.

5.2.5.4 Safety Evaluation

The leak detection system has no safety design basis; however, the containment atmosphere radioparticulate and radiogas monitors are qualified for an SSE per the recommendation of Regulatory Guide 1.45.

5.2.5.5 Tests and Inspections

Periodic testing of leakage detection systems is conducted to verify the operability and sensitivity of detector equipment. These tests include installation calibrations and alignments, periodic channel calibrations, frictional tests, and channel checks. A description of calibration and maintenance procedures and frequencies for the containment radioactivity monitoring system is presented in subsection 11.5.2.

The humidity detector and condensate measuring system are also periodically tested to ensure proper operation and verify sensitivity.

Inservice inspection criteria, the equipment used, procedures involved, the frequency of testing, inspection, surveillance, and examination of the structural and leaktight integrity of RCPB components are described in detail in subsection 5.2.4.

5.2.5.6 Instrumentation Applications

The following indications are provided in the control room to allow operating personnel to monitor for leakage:

- A. Containment air particulate monitor air particulate activity.
- B. Containment gaseous activity monitor gaseous activity.
- C. Containment cooler condensate monitoring system standpipe level.
- D. Containment humidity measuring system containment humidity.
- E. Containment normal sump level and reactor cavity sump level.
- F. Gross leakage detection methods charging pump flowrate, letdown flowrate, pressurizer level, and reactor coolant temperatures are available for the charging pump flow method. Containment sump levels and pump operation are available for the sump pump operation method. Total makeup waterflow is available from the plant computer for liquid inventory.

TABLE 5.2.1-1

APPLICABLE CODE ADDENDA FOR RCS COMPONENTS

Reactor vessel ASME III, 1971 Edition through Summer

1972 Addenda

Steam generator ASME III, 1971 Edition through Summer

1972 Addenda

Pressurizer ASME III, 1971 Edition through Summer

1972 Addenda

Control rod drive mechanism (CRDM)

housing

ASME III, 1974 Edition through Summer

1975 Addenda

CRDM head adapter ASME III, 1971 Edition through Summer

1972 Addenda

Reactor coolant pump ASME III, 1971 Edition through 1972 Winter

Addenda

Reactor coolant pipe ASME III, 1974 Edition through Winter 1975

Addenda

Surge line ASME III, 1974 Edition through Winter 1975

Addenda

Valves

Pressurizer safety ASME III, 1971 Edition through Winter 1972

Addenda

Motor operated ASME III, 1974 Edition through Summer

1974 Addenda

Manual (3 in. and larger)

ASME III, 1974 Edition through Summer

1974 Addenda

Control ASME III, 1971 Edition through Summer

1972 Addenda

TABLE 5.2.3-1 (SHEET 1 OF 5)

PRIMARY AND AUXILIARY COMPONENTS MATERIAL SPECIFICATIONS

Reactor	<u>Vessel</u>	Components

Shell and head plates (other than core region)	SA-533, Grade A, B, or C, Class 1 or 2 (vacuum treated)
Shell plates (core region)	SA-533, Grade A or B, Class 1 (vacuum treated)
Shell, flange, and nozzle forgings, nozzle safe ends	SA-508, Class 2 or 3; SA-182; Grade F304 or F316
Control rod drive mechanism (CRDM) and/or ECCS appurtenances, upper head	SB-166 or SB-167 and SA-182 Grade F304
Instrumentation tube appurtenances, lower head	SB-166 or SB-167 and SA-182, Grade F304, F304L, or F316
Closure studs, nuts, washers, inserts, and adaptors	SA-540, Class 3, Grade B23 or B24
Core support pads	SB-166 with carbon less than 0.10 percent
Monitor tubes and vent pipe	SA-312 or SA-376, Grade TP304 or TP316 or SB-166, SB-167, or SA-182, Grade F316
Vessel supports, seal ledge, and head lifting lugs	SA-516, Grade 70 (quenched and tempered) or SA-533, Grade A, B, or C, Class 1 or 2 (Vessel supports may be of weld metal buildup of equivalent strength of the nozzle material.)
Cladding and buttering	Stainless Steel Weld Metal Analysis A-8 and Ni- Cr-Fe Weld Metal F-Number 43

TABLE 5.2.3-1 (SHEET 2 OF 5)

Steam Generator Components

Pressure plates SA-533, Grade A, B, or C, Class 1 or 2

Pressure forgings (including nozzles and

tube sheet)

SA-508, Class 1, 2, 2a, or 3

Nozzle safe ends Stainless Steel Weld Metal Analysis A-8

Channel heads SA-533, Grade A, B, or C, Class 1 or 2 or SA-

216, Grade WCC

Tubes SB-163 (Ni-Cr-Fe annealed)

Cladding and buttering Stainless Steel Weld Metal Analysis A-8 and

Ni-Cr-Fe Weld Metal F-Number 43

Closure studs/nuts SA-193, Grade B7/SA-194 Gr 7

Pressurizer Components

Pressure plates SA-533, Grade A, Class 2

Pressure forgings SA-508, Class 2 or 2a

Nozzle safe ends SA-182, Grade F316L

Cladding and buttering Stainless Steel Weld Metal Analysis A-8 and

Ni-Cr-Fe Weld Metal F-Number 43

Closure studs/nuts SA-193, Grade B7/SA-194 Gr 7

Reactor Coolant Pump

Pressure forgings SA-182, Grade F304, F316, F347, or F348

Pressure casting SA-351, Grade CF8, CF8A, or CF8M

Tube and pipe SA-213; SA-376 or SA-312, Seamless, Grade

TP304 or TP316

TABLE 5.2.3-1 (SHEET 3 OF 5)

Pressure plates SA-240, Type 304 or 316

Bar material SA-479, Type 304 or 316

Closure bolting SA-193, SA-320, SA-540, or SA-453,

Grade 660

Flywheel SA-533, Grade B, Class 1

Reactor Coolant Piping

Reactor coolant pipe SA-351, Grade CF8A Centrifugal Casting

Reactor coolant fittings, branch nozzles SA-351, Grade CF8A and SA-182, (Code

Case 1423-2) Grade 316N

Surge line SA-376, Grade TP304, TP316, or F304N

Auxiliary piping SA-312 and SA-376 Grades TP304 and

TP316 to ANSI B36.10 or B36.19

Socket weld fittings ANSI B16.11

Piping flanges ANSI B16.5

Full-Length CRDM

Latch housing SA-182, Grade F304 or SA-351, Grade CF8

Rod travel housing SA-182, Grade F304 or SA-336 Class 304

Cap SA-479, Type 304

Welding materials Stainless Steel Weld Metal Analysis A-8

<u>Valves</u>

Bodies SA-182, Grade F316 or SA-351, Grade CF8 or

CF8M

TABLE 5.2.3-1 (SHEET 4 OF 5)

Bonnets SA-182, Grade F316 or SA-351, Grade CF8 or

CF8M

Discs SA-182, Grade F316 or SA-564, Grade 630 or

SA-351, Grade CF8 or CF8M

Stems SA-182, Grade F316 or SA-564, Grade 630

Pressure retaining bolting SA-453, Grade 660

Pressure retaining nuts SA-453, Grade 660 or SA-194, Grade 6

Auxiliary Heat Exchangers

Heads SA-240, Type 304

nozzle necks SA-182, Grade F304, SA-240 and 312, Type

304

Tubes SA-213, Grade TP304; SA-249 Type 304

Tube sheets SA-182, Grade F304; SA-240, Type 304 and

SA-516 GR 70 clad with Stainless Steel Weld

Metal Analysis A-8

Shells SA-240 and SA-312, Grade TP304

Auxiliary Pressure Vessels, Tanks, Filters, etc.

Shells and heads SA-240, Type 304 or SA-351 Grade CFA8, or

SA264 (consisting of SA-537, Class 1 with Stainless Steel Weld Metal Analysis A-8

Cladding)

Flanges and nozzles SA-182, Grade F304 and SA-105 or SA-350,

Grade LF2 and LF3 with Stainless Steel Weld

Metal Analysis A-8 Cladding

Piping SA-312 Type 304 and SA-240, Grade TP304

or TP316

TABLE 5.2.3-1 (SHEET 5 OF 5)

Pipe fittings SA-403, Grade WP304 Seamless

Closure bolting and nuts SA-193, Grade B7 and SA-194,

Grade 2H or Grade 7

Auxiliary Pumps

Pump casing and heads SA-351, Grade CF8 or CF8M; SA-182, Grade

F304 or F316

Flanges and nozzles SA-182, Grade F304 or F316; SA-403 Grade

WP316L Seamless

Piping SA-312, Grade TP304 or TP316 Seamless

Stuffing or packing box cover SA-351, Grade CF8 or CF8M; SA-240 Type

304 or 304L or 316

Pipe fittings SA-403, Grade WP316L Seamless

Closure bolting and nuts SA-193, Grade B6, B7, or B8M; SA-194,

Grade 2H or 8M; SA-453 Grade 660, and

Nuts, SA-194 Grade 2H, 6 and 8M

TABLE 5.2.3-2

REACTOR VESSEL INTERNALS MATERIAL SPECIFICATIONS

Forgings SA-182, Grade F304

Plates SA-240, Type 304

Pipes SA-312, Grade TP304 Seamless or SA-376,

Grade TP304

Tubes SA-213, Grade TP304 or ASTM A-511 Grade

MT304, Code Case1618

Bars SA-479, Type 304

Castings SA-351, Grade CF8

Bolting SA-193, Class 2 (65-90 YS/90 MTS) Code Case

1618 Inconel-750; SA-637, Grade 688, Type 2

Nuts SA-193, Grade B8

Locking devices SA-479, Type 304

TABLE 5.2.3-3 (SHEET 1 OF 2)

RECOMMENDED^(a) REACTOR COOLANT WATER CHEMISTRY SPECIFICATION

Electrical conductivity Determined by the concentration of boric acid and

alkali present. Expected range is <1 to 40 μmhos/cm

at 25°C.

Solution pH Determined by the concentration of boric acid and

alkali present. Expected values range between 4.2 (high boric acid concentration) and 10.5 (low boric acid concentration) at 25°C. Values will be 5.0 or

greater at normal operating temperatures.

Oxygen^(b) 0.10 ppm, maximum

Chloride^(c) 0.15 ppm, maximum

Fluoride^(c) 0.15 ppm, maximum

Hydrogen^(d) 25 to 50 cm³ (STP)/kg H₂O

Suspended solids^(e) 0.2 ppm, maximum

pH control agent (Li₇OH) Lithium is coordinated with boron in accordance with

the principles of the EPRI PWR Primary Water

Chemistry Guidelines, Volume 1.

Boric acid Variable from 0 to 4000 ppm as boron

Zinc^(g) 0.04 ppm, maximum

Silica^(f) 1.0 ppm, maximum

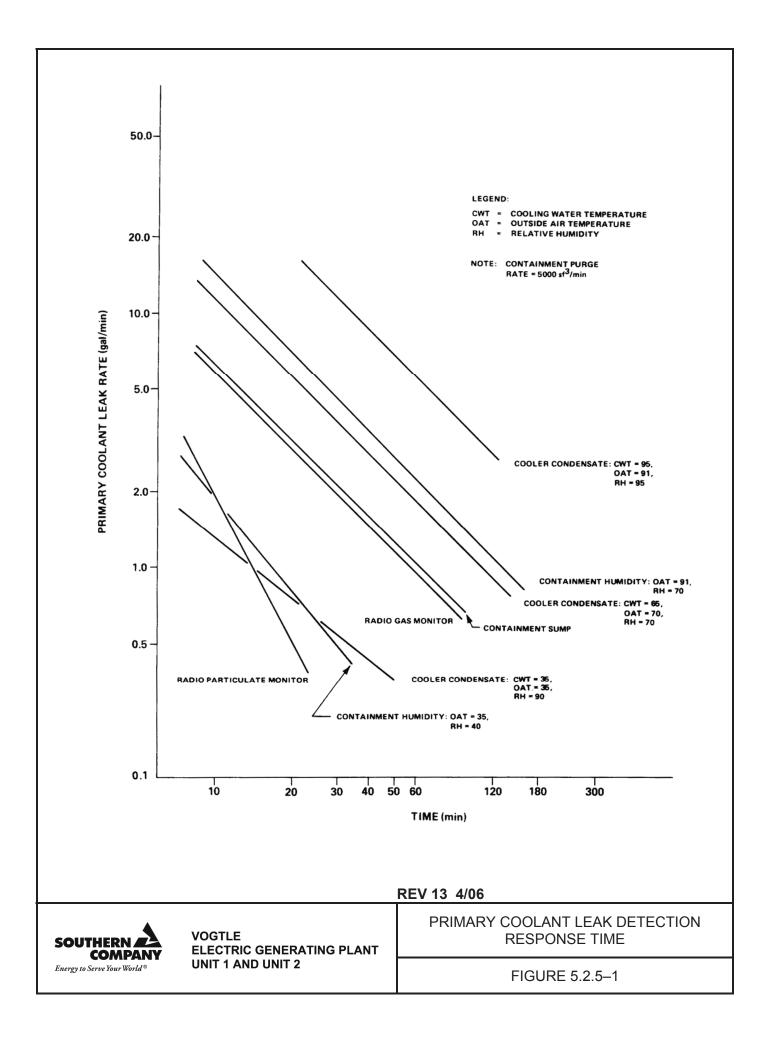
Aluminum^(f) 0.05 ppm, maximum

Calcium^(f) + magnesium 0.05 ppm, maximum

Magnesium^(f) 0.025 ppm, maximum

TABLE 5.2.3-3 (SHEET 2 OF 2)

- a. Refer to the Technical Requirements Manual for required reactor coolant chemistry limits.
- b. Oxygen concentration must be controlled to less than 0.1 ppm in the reactor coolant by scavenging with hydrazine prior to plant operation above 250°F. During power operation with the specified hydrogen concentration maintained in the coolant, the residual oxygen concentration will not exceed 0.005 ppm.
- c. Halogen concentrations must be maintained below the specified values at all times when fuel is in the reactor vessel regardless of system temperature.
- d. Hydrogen must be maintained in the reactor coolant for all plant operations with nuclear power above 1 MW.
- e. Solids concentration determined by filtration through filter having 0.45-µm pore size.
- f. These limits are included in the table of reactor coolant specifications as recommended standards for monitoring coolant purity. Establishing coolant purity within the limits shown for these species is judged desirable with the current data base to minimize fuel clad crud deposition which affects the corrosion resistance and heat transfer of the clad.
- g. Specification is applicable during power operation when zinc is being injected. Zinc target concentrations are maintained at the lower of 0.04 ppm or that specified in the reload safety analysis.



5.3 **REACTOR VESSEL**

5.3.1 REACTOR VESSEL MATERIALS

5.3.1.1 <u>Material Specifications</u>

Material specifications are in accordance with the American Society of Mechanical Engineers (ASME) Code requirements and are given in subsection 5.2.3. All ferritic reactor vessel materials comply with the fracture toughness requirements of Section 50.55a and Appendices G and H of 10 CFR 50.

The ferritic materials of the reactor vessel beltline are restricted to the following maximum limits of copper and phosphorus to reduce sensitivity to irradiation embrittlement in service:

<u>Element</u>	Base Metal (percent)	As Deposited Weld Metal (percent)
Copper	0.10 (ladle)	0.10
	0.12 (check)	
Phosphorus	0.012 (ladle)	0.020
	0.017 (check)	

5.3.1.2 Special Processes Used for Manufacturing and Fabrication

- A. The vessel is Safety Class 1. Design and fabrication of the reactor vessel is carried out in strict accordance with ASME Code, Section III, Class 1 requirements. The vessel head, flanges, and nozzles are manufactured as forgings. The cylindrical portion of the vessel is made of several shells, each consisting of formed plates joined by full penetration longitudinal and girth weld seams. The hemispherical heads are made from dished plates. The reactor vessel parts are joined by welding, using the single or multiple wire submerged arc and the shielded metal arc processes.
- B. The use of severely sensitized stainless steel as a pressure boundary material has been prohibited and has been eliminated by either a select choice of material or by programming the method of assembly.
- C. The control rod drive mechanism (CRDM) head adapter threads and surfaces of the guide studs are chrome plated to prevent possible galling of the mated parts.
- D. At all locations in the reactor vessel where stainless steel and Inconel are joined, the final joining beads are Inconel weld metal in order to prevent cracking.
- E. The location of full penetration weld seams in the upper closure head and vessel bottom head are restricted to areas that permit accessibility during inservice inspection.
- F. The stainless steel clad surfaces are sampled to ensure that composition requirements are met.

- G. Freedom from underclad cracking is ensured by special evaluation of the procedure qualification for cladding applied on low-alloy steel (SA-508, Class 2).^a
- H. Minimum preheat requirements have been established for pressure boundary welds using low-alloy material. The preheat is maintained until either an intermediate postweld heat treatment or a full postweld heat treatment is completed or until the completion of welding.
- I. A field weld is made, after the reactor vessel has been set, to install the permanent reactor vessel cavity seal ring. This stainless steel filler weld joins the seal ring to the reactor vessel seal ledge. A minimum preheat is specified for this weld in compliance with the ASME Code requirements.

5.3.1.3 Special Methods for Nondestructive Examination

The nondestructive examination (NDE) of the reactor vessel and its appurtenances is conducted in accordance with ASME Code, Section III requirements; also, numerous examinations are performed in addition to ASME Code, Section III requirements. The NDE of the vessel is discussed in the following paragraphs, and the reactor vessel quality assurance program is given in table 5.3.1-1.

5.3.1.3.1 Ultrasonic Examination

- A. In addition to the required ASME Code straight beam ultrasonic examination, angle beam inspection over 100 percent of one major surface of plate material is performed during fabrication to detect discontinuities that may be undetected by the straight beam examination.
- B. In addition to the ASME Code, Section III NDE, all full penetration ferritic pressure boundary welds in the reactor vessel are ultrasonically examined during fabrication. This test is performed upon completion of the welding and intermediate heat treatment but prior to the final postweld heat treatment.
- C. After hydrotesting, all full penetration ferritic pressure boundary welds in the reactor vessel, as well as the nozzle to safe end welds, are ultrasonically examined. These inspections are also performed in addition to the ASME Code, Section III NDE.

5.3.1.3.2 Penetrant Examinations

The partial penetration welds for the CRDM head adapters and the bottom instrumentation tubes are inspected by dye penetrant after the root pass, in addition to code requirements. Core support block attachment welds are inspected by dye penetrant after the first layer of weld metal and after each 1/2 in. of weld metal. All clad surfaces and other vessel and head internal surfaces are inspected by dye penetrant after the hydrostatic test.

5.3-2

^a Underclad cracking of the reactor pressure vessel was evaluated as a time-limited aging analysis (TLAA) for license renewal in accordance with 10 CFR Part 54. The results of this evaluation are provided in paragraph 19.4.6.5.

5.3.1.3.3 Magnetic Particle Examination

The magnetic particle examination requirements below are in addition to the magnetic particle examination requirements of Section III of the ASME Code.

All magnetic particle examinations of materials and welds are performed in accordance with the following:

- Prior to the final postweld heat treatment, only by the prod, coil, or direct contact method.
- After the final postweld heat treatment, only by the yoke method.

The following surfaces and welds are examined by magnetic particle methods. The acceptance standards are in accordance with Section III of the ASME Code.

A. Surface Examinations

- 1. Magnetic particle examination of all exterior vessel and head surfaces after the hydrostatic test.
- 2. Magnetic particle examination of all exterior closure stud surfaces and all nut surfaces after final machining or rolling. Continuous circular and longitudinal magnetization is used.
- 3. Magnetic particle examination of all inside diameter surfaces of carbon and low alloy steel products that have their properties enhanced by accelerated cooling. This inspection is performed after forming and machining and prior to cladding.

B. Weld Examination

Magnetic particle examination of the welds attaching the closure head lifting lugs and refueling seal ledge to the reactor vessel after the first layer and each 1/2 in. of weld metal is deposited. All pressure boundary welds are examined after back-chipping or back-grinding operations.

5.3.1.4 Special Controls for Ferritic and Austenitic Stainless Steels

Welding of ferrite steels and austenitic stainless steels is discussed in subsection 5.2.3. Subsection 5.2.3 includes discussions which indicate the degree of conformance with Regulatory Guide 1.44. Section 1.9 discusses the degree of conformance with Regulatory Guides 1.43, 1.50, 1.71, and 1.99.

5.3.1.5 Fracture Toughness^a

Assurance of adequate fracture toughness of ferritic materials in the reactor vessel (ASME Code, Section III, Class 1 component) is provided by compliance with the requirements for fracture toughness testing included in NB-2300 to Section III of the ASME Code and Appendix G of 10 CFR 50.

The initial Charpy V-notch minimum upper shelf fracture energy levels for the reactor vessel beltline (including welds) are 75 ft-lb, as required by Appendix G of 10 CFR 50. The vessel

^a Reactor vessel neutron embrittlement was evaluated as a TLAA for license renewal in accordance with 10 CFR 54.21 (see subsection 19.4.1).

fracture toughness data for Units 1 and 2 are given in tables 5.3.1-2 and 5.3.1-3, respectively. The end-of-life RT_{NDT} and upper shelf energy projections estimated using Regulatory Guide 1.99 for the end-of-life neutron fluence at the 1/4 T and ID reactor vessel locations for Units 1 and 2 are given in tables 5.3.3-2 and 5.3.3-3.

5.3.1.6 Material Surveillance^a

The reactor vessel material irradiation surveillance specimens shall be removed and examined to determine changes in material properties as required by 10 CFR Part 50, Appendix H, in accordance with the schedule in FSAR tables 5.3.1-8 and 5.3.1-9. The results of these examinations shall be used to update figures in the Pressure Temperature Limits Report (PTLR).⁽¹⁶⁾ In the surveillance program, the evaluation of radiation damage is based on preirradiation testing of Charpy V-notch and tensile specimens and post irradiation testing of Charpy V-notch, tensile, and 1/2-T compact tension (CT) fracture mechanics test specimens. The program is directed toward evaluation of the effect of radiation on the fracture toughness of reactor vessel steels based on the transition temperature approach and the fracture mechanics approach. The program conforms to American Society of Testing Materials (ASTM) E-185-82, Conducting Surveillance Tests for Light-Water-Cooled Nuclear Reactor Vessels, and 10 CFR 50, Appendix H.

[HISTORICAL] The reactor vessel surveillance program uses six specimen capsules. The capsules are located in guide baskets welded to the outside of the neutron shield pads and positioned directly opposite the center portion of the core. The capsules can be removed when the vessel head is removed and can be replaced when the internals are removed. The six capsules contain reactor vessel steel specimens, oriented both parallel and normal (longitudinal and transverse) to the principal rolling direction of the limiting base material located in the core region of the reactor vessel and associated weld metal and weld heat-affected zone metal. The six capsules contain 54 tensile specimens, 360 Charpy V-notch specimens (which include weld metal and weld heat-affected zone material), and 72 CT specimens. Archive material sufficient for two additional capsules is retained at Westinghouse. The surveillance program withdrawal schedule, lead factor, test samples, and materials in the reactor vessel are given in tables 5.3.1-7 and 5.3.1-8. [HISTORICAL]

Removal of these specimen capsules was completed for Unit 1 at refueling outage 1R14 and for Unit 2 at refueling outage 2R14. Refer to paragraph 5.3.1.6.1.4 for a description of the external neutron monitoring system also known as external vessel neutron dosimetry system (EVNDS) used following removal of the last specimen capsules.

Dosimeters, as described below, are placed in filler blocks drilled to contain them. The dosimeters permit evaluation of the flux seen by the specimens and the vessel wall. In addition, thermal monitors made of low melting point alloys are included to monitor the maximum temperature of the specimens. The specimens are enclosed in a tight-fitting stainless steel sheath to prevent corrosion and ensure good thermal conductivity. The complete capsule is helium leak tested. As part of the surveillance program, a report of the residual elements in weight percent to the nearest 0.01 percent is made for surveillance material and as deposited weld metal. Each of the six capsules contains the following specimens:

5.3-4

^a The Reactor Vessel Surveillance Program is credited as a license renewal aging management program (see subsection 19.2.25).

<u>Material</u>	Number of <u>Charpys</u>	Number of <u>Tensiles</u>	Number of <u>CTs</u>
Limiting base materiala	15	3	4
Limiting base material ^b	15	3	4
Weld metal ^(c)	15	3	4
Heat-affected zone	15	_	_

The following dosimeters and thermal monitors are included in each of the six capsules:

A. Dosimeters

- 1. Iron.
- 2. Copper.
- 3. Nickel.
- 4. Cobalt-aluminum (0.15-percent cobalt).
- 5. Cobalt-aluminum (cadmium shielded).
- 6. Uranium-238 (cadmium shielded).
- 7. Neptunium-237 (cadmium shielded).

B. Thermal Monitors

- 1. 97.5-percent lead, 2.5-percent silver, (579°F melting point).
- 2. 97.5-percent lead, 1.75-percent silver, 0.75-percent tin (590°F melting point).

The fast neutron exposure of the specimens occurs at a faster rate than that experienced by the vessel wall, with the specimens being located between the core and the vessel. Since these specimens experience accelerated exposure and are actual samples from the materials used in the vessel, the transition temperature shift measurements are representative of the vessel at a later time in life. Data from CT fracture toughness specimens are expected to provide additional information for use in determining allowable stresses for irradiated material.

Correlations between the calculations and measurements of the irradiated samples in the capsules, assuming the same neutron spectrum at the samples and the vessel inner wall, are described in paragraph 5.3.1.6.1. The anticipated degree to which the specimens perturb the fast neutron flux and energy distribution is considered in the evaluation of the surveillance specimen data. Verification and possible readjustment of the calculated wall exposure is made by the use of data on all capsules withdrawn. The schedule for removal of the capsules for postirradiation testing conforms with ASTM E-185-82 and Appendix H of 10 CFR 50.

5.3.1.6.1 Measurement of Integrated Fast Neutron (E > 1.0 MeV) Flux at the Irradiation Samples

The use of passive neutron sensors such as those included in the internal surveillance capsule dosimetry sets does not yield a direct measure of the energy dependent neutron flux level at the

^a Specimens oriented in the major rolling or working direction.

^b Specimens oriented normal to the major rolling or working direction.

⁽c) Weld metal to be selected in accordance with ASTM E-185-82.

measurement location. Rather, the activation or fission process is a measure of the integrated effect that the time- and energy-dependent neutron flux has on the target material over the course of the irradiation period. An accurate assessment of the average flux level and, hence, time integrated exposure (fluence) experienced by the sensors may be developed from the measurements only if the sensor characteristics and the parameters of the irradiation are well known. In particular, the following variables are of interest:

- The measured specific activity of each sensor.
- The physical characteristics of each sensor.
- The operating history of the reactor.
- The energy response of each sensor.
- The neutron energy spectrum at the sensor location.

This section describes the procedures used to determine sensor specific activities, to develop reaction rates for individual sensors from the measured specific activities and the operating history of the reactor, and to derive key fast neutron exposure parameters from the measured reaction rates.

5.3.1.6.1.1 <u>Determination of Sensor Reaction Rates.</u> The specific activity of each of the radiometric sensors is determined using established ASTM procedures. Following sample preparation and weighing, the specific activity of each sensor is determined by means of a high purity germanium gamma spectrometer. In the case of the surveillance capsule multiple foil sensor sets, these analyses are performed by direct counting of each of the individual wires or, as in the case of U-238 and Np-237 fission monitors, by direct counting preceded by dissolution and chemical separation of cesium from the sensor.

The irradiation history of the reactor over its operating lifetime is determined from plant power generation records. In particular, operating data are extracted on a monthly basis from reactor startup to the end of the capsule irradiation period. For the sensor sets utilized in the surveillance capsule irradiations, the half-lives of the product isotopes are long enough that a monthly histogram describing reactor operation has proven to be an adequate representation for use in radioactive decay corrections for the reactions of interest in the exposure evaluations.

Having the measured specific activities, the operating history of the reactor, and the physical characteristics of the sensors, reaction rates referenced to full power operation are determined from the following equation:

$$R = \frac{A}{N_0 F Y \sum_{j} \frac{P_j}{P_{ref}} C_j \left[1 - e^{-\lambda t_j} \right] e^{-\lambda t_d}}$$

where:

A = measured specific activity provided in terms of disintegrations per second per gram of target material (dps/grn).

R = reaction rate averaged over the irradiation period and referenced to operation at a core power level of P_{ref} expressed in terms of reactions per second per nucleus of target

isotope (rps/nucleus).

N₀ = number of target element atoms per gram of sensor.

Selisor.

F = weight fraction of the target isotope in the

sensor material.

Y = number of product atoms produced per

reaction.

P_j = average core power level during irradiation

period j (MW).

P_{ref} = maximum or reference core power level of the

reactor (MW).

 C_j = calculated ratio of $\phi(E > 1.0 \text{ MeV})$ during irradiation period j to the time weighted average

 $\phi(E > 1.0 \text{ MeV})$ over the entire irradiation

period.

 λ = decay constant of the product isotope (sec⁻¹).

t_i = length of irradiation period j (sec).

t_d = decay time following irradiation period j (sec).

and the summation is carried out over the total number of monthly intervals comprising the total irradiation period.

In the above equation, the ratio P_j/P_{ref} accounts for month-by-month variation of power level within a given fuel cycle. The ratio C_j is calculated for each fuel cycle and accounts for the change in sensor reaction rates caused by variations in flux level due to changes in core power spatial distributions from fuel cycle to fuel cycle. Since the neutron flux at the measurement locations within the surveillance capsules is dominated by neutrons produced in the peripheral fuel assemblies, the change in the relative power in these assemblies from fuel cycle to fuel cycle can have a significant impact on the activation of neutron sensors. For a single-cycle irradiation, $C_j = 1.0$. However, for multiple-cycle irradiations, particularly those employing low-leakage fuel management, the additional C_j correction must be utilized in order to provide accurate determinations of the decay-corrected reaction rates for the dosimeter sets contained in the surveillance capsules.

5.3.1.6.1.2 <u>Corrections to Reaction Rate Data.</u> Prior to using the measured reaction rates in the least squares adjustment procedure discussed in paragraph 5.3.1.6.1.3, additional corrections are made to the U-238 measurements to account for the presence of U-235 impurities in the sensors as well as to adjust for the build-in of plutonium isotopes over the course of the irradiation.

In addition to the corrections made for the presence of U-235 in the U-238 fission sensors, corrections are also made to both the U-238 and Np-237 sensor reaction rates to account for gamma ray induced fission reactions occurring over the course of the irradiation.

5.3.1.6.1.3 <u>Least Squares Adjustment Procedure.</u> Least squares adjustment methods provide the capability of combining the measurement data with the neutron transport calculation resulting in a Best Estimate neutron energy spectrum with associated uncertainties. Best Estimates for key exposure parameters such as neutron fluence (E > 1.0 MeV) or iron atom displacements (dpa) along with their uncertainties are then easily obtained from the adjusted spectrum. The use of measurements in combination with the analytical results reduces the uncertainty in the calculated spectrum and acts to remove biases that may be present in the analytical technique.

In general, the least squares methods, as applied to pressure vessel fluence evaluations, act to reconcile the measured sensor reaction rate data, dosimetry reaction cross-sections, and the calculated neutron energy spectrum within their respective uncertainties. For example,

$$R_i \pm \delta_{Ri} = \sum_{g} \left(\sigma_{ig} \pm \delta_{\sigma_{ig}} \right) \left(\phi_g \pm \delta_{\phi_g} \right)$$

relates a set of measured reaction rates, R_i , to a single neutron spectrum, Φg , through the multigroup dosimeter reaction cross-section, σ_{iq} , each with an uncertainty δ .

The use of least squares adjustment methods in light water reactor (LWR) dosimetry evaluations is not new. ASTM has addressed the use of adjustment codes in ASTM Standard E944, "Application of Neutron Spectrum Adjustment Methods in Reactor Surveillance," and many industry workshops have been held to discuss the various applications. For example, the ASTM-EURATOM Symposia on Reactor Dosimetry holds workshops on neutron spectrum unfolding and adjustment techniques at each of its biannual conferences.

The primary objective of the least squares evaluation is to produce unbiased estimates of the neutron exposure parameters at the location of the measurement. The analytical method alone may be deficient because it inherently contains uncertainty due to the input assumptions to the calculation. Typically these assumptions include parameters such as the temperature of the water in the peripheral fuel assemblies, bypass region and downcomer regions, component dimensions, and peripheral core source. Industry consensus indicates that the use of the calculation alone results in overall uncertainties in the neutron exposure parameters in the range of 15-20% (1σ).

By combining the calculated results with available measurements, the uncertainties associated with the key neutron exposure parameters can be reduced. Specifically ASTM Standard E 944 states, "The algorithms of the adjustment codes tend to decrease the variances of the adjusted data compared to the corresponding input values. The least squares adjustment codes yield estimates for the output data with minimum variances, that is, the "best estimates." This is the primary reason for using these adjustment procedures." ASTM E 944 provides a comprehensive listing of available adjustment codes.

The FERRET least squares adjustment code⁽¹⁾ was initially developed at the Hanford Engineering Development Laboratory (HEDL) and has had extensive use in both the Liquid Metal Fast Breeder (LMFBR) program and the NRC Sponsored Light Water Reactor Dosimetry Improvement Program (LWR-PV-SDIP). As a result of participation in several cooperative efforts associated with the LWR-PV-SDIP, the FERRET approach was adopted by Westinghouse in the mid 1980's as the preferred approach for the evaluation of LWR surveillance dosimetry. The least squares methodology was judged superior to the previously employed spectrum averaged cross-section approach that is totally dependent on the accuracy of the shape of the calculated neutron spectrum at the measurement locations.

The FERRET code is employed to combine the results of plant specific neutron transport calculations and multiple-foil reaction-rate measurements to determine best estimate values of exposure parameters in terms of both neutron fluence greater than 1.0 MeV, $(\Phi(E > 1.0 \text{ MeV}))$

and iron atom displacements, (dpa), along with associated uncertainties in the measurement locations.

The application of the least squares methodology requires the following input:

- The calculated neutron energy spectrum and associated uncertainties at the measurement location.
- The measured reaction rate and associated uncertainty for each sensor contained in the multiple foil set.
- The energy dependent dosimetry reaction cross-sections and associated uncertainties for each sensor contained in the multiple foil sensor set.

For a given application, the calculated neutron spectrum is obtained from the results of plant-specific neutron transport calculations applicable to the irradiation period experienced by the dosimetry sensor set. This calculation is performed using the benchmarked transport calculational methodology described in paragraph 5.3.1.6.2. The sensor reaction rates are derived from the measured specific activities obtained from the counting laboratory using the specific irradiation history of the sensor set to perform the radioactive decay corrections. The dosimetry reaction cross-sections and uncertainties are obtained from the SNLRML dosimetry cross-section library⁽²⁾. The SNLRML library is an evaluated dosimetry reaction cross-section compilation recommended for use in LWR evaluations by ASTM Standard E1018, "Application of ASTM Evaluated Cross-Section Data File, Matrix E 706 (IIB)." There are no additional data or data libraries built into the FERRET code system. All of the required input is supplied externally at the time of the analysis.

The uncertainties associated with the measured reaction rates, dosimetry cross-sections, and calculated neutron spectrum are input to the least squares procedure in the form of variances and covariances. The assignment of the input uncertainties also follows the guidance provided in ASTM Standard E 944.

5.3.1.6.1.4 External Neutron Monitoring System. The external neutron monitoring system also known as the external vessel neutron dosimetry system (EVNDS) provides for continuing neutron fluence measurement after sufficient specimen material exposure has been achieved and even after the last of the six internal surveillance capsules has been removed from the reactor vessel. It enables verification of fast neutron exposure distributions within the reactor vessel wall beltline region and establishes a mechanism to enable long term monitoring of this portion of the reactor vessel as required per 10 CFR 50 Appendix H. These fluence data can also support potential license renewal activities.

The EVNDS is located external to the reactor vessel, allowing for ease of dosimetry removal and replacement. It is installed in the annular air gap between the reactor vessel insulation and the primary concrete shield wall. The EVNDS is a passive system consisting of six aluminum dosimeter capsules containing radiometric monitors and four stainless steel gradient chains, which are bead chains connecting and supporting the dosimeter capsules. The bead chains are in turn supported by an arrangement of stainless steel hardware-tubular brackets on a support bar suspended by chains from bracket plate assembly, which is welded to the ventilation port liner plate under a banana cover. The bead chains are mechanically secured to the concrete floor below the reactor vessel. The system is shown on drawings 1X6AB03-00020, 1X6AB03-00021, and 1X6AB03-00022 for Unit 1 and is shown on drawings 2X6AB03-00022, 2X6AB03-00023, and 2X6AB03-00024 for Unit 2.

The EVNDS measures fluence for approximately 1/8 of the vessel wall circumference, positioned relative to well known reactor features. Neutron transport calculations then determine the fluence for the entire vessel beltline wall. The system assists in the evaluation of radiation damage to the reactor vessel beltline region by measuring the fluence to this region, which can be used to predict the shift in the reference nil ductility transition temperature (RT_{NDT}). When used in conjunction with previously removed dosimetry from the internal surveillance capsules and with the results of neutron transport calculations, the external vessel neutron measurements allow the projection of embrittlement gradients through the reactor vessel wall with minimum uncertainty. Minimizing the uncertainty in the neutron exposure projections will help to assure that the reactor can be operated in the least restrictive mode possible with respect to:

- 10 CFR 50 Appendix G pressure / temperature limit curves for normal heatup and cooldown of the reactor coolant system,
- Emergency Response Guideline (ERG) pressure / temperature limit curves, and
- Pressurized thermal shock (PTS) RT_{NDT} screening criteria.

Comprehensive sensor sets are employed at discrete locations within the reactor cavity to characterize the neutron energy spectrum variations axially and azimuthally over the beltline region of the reactor vessel. In addition, the stainless steel gradient chains are used in conjunction with the encapsulated dosimeters to complete the mapping of the neutron environment between the discrete locations chosen for spectrum determinations.

The first replacement of irradiation dosimetry is at refueling outage 1R15 (Sept. 2009). The first replacement of irradiation dosimetry is at refueling outage 2R14 (March 2010). An irradiation interval of five fuel cycles between replacements is typical.

5.3.1.6.2 Calculation of Integrated Fast (E > 1.0 MeV) Exposure at the Irradiation Samples and Reactor Vessel Wall

Discrete ordinates transport calculations are performed on a fuel cycle-specific basis to determine the neutron and gamma ray environment within the reactor geometry. The specific methods applied have been benchmarked according to the guidelines of Regulatory Guide 1.190, "Calculational and Dosimetry Methods for Determining Pressure Vessel Neutron Fluence," U. S. Nuclear Regulatory Commission, Office of Nuclear Regulatory Research, March 2001, and have been approved by the NRC staff for general application to pressurized water reactor (PWR) analysis. A description of the transport methodology along with the SER documenting NRC staff approval of the method and computer codes are provided in Reference 13.

In the application of this methodology to the fast neutron exposure evaluations for the surveillance capsules and reactor vessel, a series of two-dimensional plant-specific transport calculations are carried out and then synthesized to generate a three-dimensional neutron flux distribution, $\Phi(r,\theta,z)$ throughout the geometry of interest using the procedures outlined in Regulatory Guide 1.190. These three-dimensional mappings of the neutron environment are completed for each operating fuel cycle and then integrated to determine the neutron fluence experienced by the surveillance test specimens and the pressure vessel wall.

In the approved analysis methodology, the transport calculations are completed using the DORT discrete ordinates code Version 3.1⁽³⁾ and the BUGLE-96 cross-section library⁽¹⁰⁾. The BUGLE-96 library provides a 67 group coupled neutron-gamma ray cross-section data set

produced specifically for LWR application. In these analyses, anisotropic scattering is treated with a P_5 legendre expansion and the angular discretization is modeled with an S_{16} order of angular quadrature.

Energy- and space-dependent core power distributions, as well as system operating temperatures, are treated on a fuel cycle-specific basis. The spatial variation of the neutron source is obtained from a burnup-weighted average of the respective power distributions from individual fuel cycles including pinwise gradients for all fuel assemblies located on the periphery of the core. The energy distribution of the source is determined on a fuel assembly-specific basis and includes the effects of fissioning in both uranium and plutonium isotopes.

The results of the transport calculations are validated on a plant-specific basis by comparison with the results of surveillance capsule dosimetry developed using the procedures described in paragraph 5.3.1.6.1. These comparisons are used to demonstrate that the plant-specific application is consistent with the uncertainty evaluations provided in Reference 13 and to establish that the 20% uncertainty criterion listed in Regulatory Guide 1.190 is met. These comparisons are not used to modify or bias the results of the transport calculations.

5.3.1.6.2.1 Reference Forward Calculation. The forward transport calculation for the reactor is carried out in r,0 geometry using the DORT two-dimensional discrete ordinates $code^{(3)}$ and the BUGLE-96 cross-section library⁽¹⁰⁾. The BUGLE-96 library is a 47 neutron group, ENDFB-VI based, data set produced specifically for LWR applications. In these analyses, anisotropic scattering is treated with a P_3 expansion of the scattering cross-sections and the angular discretization is modeled with an S_8 order of angular quadrature. The reference forward calculation is normalized to a core midplane power density characteristic of operation at the stretch rating for the reactor.

The spatial core power distribution utilized in the reference forward calculation is derived from statistical studies of long-term operation of Westinghouse 4-loop plants. Inherent in the development of this reference core power distribution is the use of an out-in fuel management strategy; i.e., fresh fuel on the core periphery. Furthermore, a 2σ uncertainty derived from the statistical evaluation of plant-to-plant and cycle-to-cycle variations in peripheral power is used for the peripheral fuel assemblies.

Due to the use of this bounding spatial power distribution, the results from the reference forward calculation establish conservative exposure projections for reactors of this design operating at the stretch rating. Since it is unlikely that actual reactor operation would result in the implementation of a power distribution at the nominal $+2\sigma$ level for a large number of fuel cycles and, further, because of the widespread implementation of low-leakage fuel management strategies, the fuel cycle-specific calculations for this reactor will result in exposure rates well below these conservative predictions.

5.3.1.6.2.2 Cycle-Specific Adjoint Calculations. All adjoint analyses are also carried out using an S_8 order of angular quadrature and the P_3 cross-section approximation from the BUGLE-96 library. Adjoint source locations are chosen at several key azimuths on the pressure vessel inner radius. In addition, adjoint calculations were carried out for sources positioned at the geometric center of all surveillance capsules. Again, these calculations are run in r,θ geometry to provide neutron source distribution importance functions for the exposure parameter of interest; in this case, $\phi(E > 1.0 \text{ MeV})$.

The importance functions generated from these individual adjoint analyses provide the basis for all absolute exposure projections and comparison with measurement. These importance functions, when combined with cycle-specific neutron source distributions, yield absolute predictions of neutron exposure at the locations of interest for each of the operating fuel cycles and establish the means to perform similar predictions and dosimetry evaluations for all subsequent fuel cycles.

Having the importance functions and appropriate core source distributions, the response of interest can be calculated as:

$$\varphi(R_o, \theta_o) = \int_r \int_{\theta} \int_{E} I(r, \theta, E) S(r, \theta, E) r dr d\theta dE$$

where: $\varphi(R_0, \theta_0)$ = Neutron flux (E > 1.0 MeV) at radius R_0 and azimuthal angle θ_0 .

 $I(r,\theta,E)$ = Adjoint importance function at radius r, azimuthal angle θ , and neutron source energy E.

 $S(r,\theta,E)$ = Neutron source strength at core location r,θ and energy E.

It is important to note that the cycle-specific neutron source distributions, $S(r,\theta,E)$, utilized with the adjoint importance functions, $I(r,\theta,E)$, permit the use not only of fuel cycle-specific spatial variations of fission rates within the reactor core, but also allow for the inclusion of the effects of the differing neutron yield per fission and the variation in fission spectrum introduced by the build-in of plutonium isotopes, as the burnup of individual fuel assemblies increases.

5.3.1.7 <u>Reactor Vessel Fasteners</u>

The reactor vessel closure studs, nuts, and washers are designed and fabricated in accordance with the requirements of the ASME Code, Section III. The closure studs are fabricated of SA-540, Class 3, Grade B24. The closure stud material meets the fracture toughness requirements of the ASME Code, Section III, and 10 CFR 50, Appendix G. Conformance with Regulatory Guide 1.65, Materials and Inspections for Reactor Vessel Closure Studs, is discussed in section 1.9. Nondestructive examinations are performed in accordance with the ASME Code, Section III. Bolting material properties for Units 1 and 2 are given in tables 5.3.1-4 and 5.3.1-5, respectively.

Refueling procedures require that the reactor vessel studs, nuts, and washers are lifted part of the way out of their respective holes and a stud support collar be put in place prior to the lift of the integrated head assembly during preparation for refueling. In this way the studs are lifted with and stored on the head. An alternative method of the procedures is that the reactor vessel studs, nuts, and washers may be removed from the reactor closure and placed in storage racks during preparation for refueling. In this method, the storage racks are removed from the refueling cavity and stored at convenient locations on the containment operating deck prior to removal of the reactor closure head and refueling cavity flooding. In either case, the reactor closure studs are not exposed to the borated refueling cavity water. Additional protection against the possibility of incurring corrosion effects is ensured by the use of a manganese base phosphate surfacing treatment.

The stud holes in the reactor flange are sealed with special plugs before removing the reactor closure, thus preventing leakage of the borated refueling water into the stud holes.

5.3.1.8 References

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5.3.2 PRESSURE-TEMPERATURE LIMITS

5.3.2.1 Limit Curves

Startup and shutdown operating limitations are based on the properties of the reactor pressure vessel beltline materials. Actual material property test data are used. The methods outlined in Appendix G Section XI of the American Society of Mechanical Engineers (ASME) Code are employed for the shell regions in the analysis of protection against nonductile failure. The initial operating curves are calculated, assuming a period of reactor operation such that the beltline material will be limiting. The heatup and cooldown curves are given in the Pressure and Temperature Limits Report as required by the Technical Specifications. Beltline material properties degrade with radiation exposure, and this degradation is measured in terms of the adjusted reference nil ductility temperature, which includes a reference nil ductility temperature shift (ΔRT_{NDT}). The reference temperature, RT_{NDT}, for materials in the reactor vessel closure flange region and the beltline regions are shown in tables 5.3.1-2 and 5.3.1-3. Tables 5.3.2-2 through 5.3.2-5 give the properties for the vessel beltline materials and data for the C_v curve (energy vs. temperature).

Predicted ΔRT_{NDT} values are derived using guidance provided in Regulatory Guide 1.99 Revision 2. For a selected time of operation, this shift is assigned a sufficient magnitude so that no unirradiated ferritic materials in other components of the reactor coolant system (RCS) will be limiting in the analysis.

The operating curves including pressure-temperature limitations shown in the Pressure and Temperature Limits Report are calculated in accordance with 10 CFR 50, Appendices G and H, and ASME Code, Section XI, Appendix G requirements, Code Case N-640 and WCAP-16142, "Reactor Vessel Closure Head/Vessel Flange Requirements Evaluation for Vogtle Units 1 and 2, Revision 1."

The results of the material surveillance program described in paragraph 5.3.1.6 will be used to verify that the ΔRT_{NDT} predicted from the effects of the fluence, copper content curve is appropriate if ΔRT_{NDT} determined from the surveillance program is greater than the predicted ΔRT_{NDT} . Temperature limits for inservice leak and hydrotests will be calculated in accordance with ASME Code, Section XI, Appendix G Conformance with Regulatory Guide 1.99, Revision 2, is discussed in section 1.9.

5.3.2.2 Operating Procedures

The transient conditions that are considered in the design of the reactor vessel are presented in paragraph 3.9.1.1. These transients are representative of the operating conditions that should prudently be considered to occur during plant operation. The transients selected form a conservative basis for evaluation of the RCS to ensure the integrity of the RCS equipment.

Those transients listed as upset condition transients are given in table 3.9.N.1-1. None of these transients will result in pressure-temperature changes which exceed the heatup and cooldown limitations, as described in the Pressure and Temperature Limits Report.

5.3.3 REACTOR VESSEL INTEGRITY

5.3.3.1 Design

The reactor vessel is cylindrical with a welded hemispherical bottom head and a removable, bolted, flanged, and gasketed hemispherical upper head. The reactor vessel flange and head are sealed by two hollow metallic O-rings. Seal leakage is detected by means of two leakoff connections, one between the inner and outer ring and one outside the outer O-ring. The vessel contains the core, core support structures, control rods, and other parts directly associated with the core. The reactor vessel closure head contains head adapters. These head adapters are tubular members, attached by partial penetration welds to the underside of the closure head. The upper end of these adapters contains acme threads for the assembly of control rod drive mechanisms (CRDMs) or instrumentation adapters. The seal arrangement at the upper end of these adapters consists of a welded flexible canopy seal. Mechanical assemblies may be used to fix or prevent leaks in the canopy seal weld. Inlet and outlet nozzles are located symmetrically around the vessel. Outlet nozzles are arranged on the vessel to facilitate optimum layout of the reactor coolant system equipment. The inlet nozzles are tapered from the coolant loop vessel interfaces to the vessel inside wall to reduce loop pressure drop.

The bottom head of the vessel contains penetration nozzles for connection and entry of the nuclear incore instrumentation. Each nozzle consists of a tubular member made of either an Inconel or an Inconel-stainless steel composite tube. Each tube is attached to the inside of the bottom head by a partial penetration weld.

Internal surfaces of the vessel which are in contact with primary coolant are weld overlay with 0.125-in. minimum of stainless steel or Inconel.

The reactor vessel is designed and fabricated in accordance with the requirements of the American Society of Mechanical Engineers (ASME) Code, Section III. Principal design parameters of the reactor vessel are given in table 5.3.3-1. The reactor vessel is shown in figure 5.3.3-1.

There are no special design features which would prohibit the <u>in situ</u> annealing of the vessel. If the unlikely need for an annealing operation was required to restore the properties of the vessel material opposite the reactor core because of neutron irradiation damage, a metal temperature greater than 650 °F for a maximum period of 168 h would be applied.⁽⁴⁾ Various modes of heating may be used, depending on the temperature required.

The reactor vessel materials surveillance program is adequate to accommodate the annealing of the reactor vessel. Sufficient specimens are available to evaluate the effects of the annealing treatment.

Cyclic loads are introduced by normal power changes, reactor trips, and startup and shutdown operations. These design base cycles are selected for fatigue evaluation and constitute a conservative design envelope for the projected plant life^a. Vessel analysis results in a usage factor that is less than 1.

The design specifications require analysis to prove that the vessel is in compliance with the fatigue and stress limits of the ASME Code, Section III. The loadings and transients specified for the analysis are based on the most severe conditions expected during service. The heatup

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^a Metal fatigue is evaluated as a TLAA for license renewal (see subsection 19.4.2).

and cooldown rates are 100°F/h for normal operations and under abnormal or emergency conditions. This rate is reflected in the vessel design specifications.

5.3.3.2 <u>Materials of Construction</u>

The materials used in the fabrication of the reactor vessel are discussed in subsection 5.2.3.

5.3.3.3 Fabrication Methods

The VEGP reactor vessel manufacturer is Combustion Engineering Corporation.

The fabrication methods used in the construction of the reactor vessel are discussed in paragraph 5.3.1.2.

5.3.3.4 <u>Inspection Requirements</u>

The nondestructive examinations performed on the reactor vessel are described in paragraph 5.3.1.3.

5.3.3.5 Shipment and Installation

The reactor vessel is shipped in a horizontal position on a shipping sled with a vessel-lifting truss assembly. All vessel openings are sealed to prevent the entrance of moisture, and an adequate quantity of desiccant bags is placed inside the vessel. These are usually placed in a wire mesh basket attached to the vessel cover. All carbon steel surfaces, except for the vessel support surfaces and the top surface of the external seal ring, are painted with a heat-resistant paint before shipment.

The closure head is also shipped with a shipping cover and skid. An enclosure attached to the ventilation shroud support ring protects the control rod mechanism housings. All head openings are sealed to prevent the entrance of moisture, and an adequate quantity of desiccant bags is placed inside the head. These are placed in a wire mesh basket attached to the head cover. All carbon steel surfaces are painted with heat- resistant paint before shipment. A lifting frame is provided for handling the vessel head.

5.3.3.6 **Operating Conditions**

Operating limitations for the reactor vessel are presented in the Pressure Temperature Limit Report (PTLR).

In addition to the analysis of primary components discussed in paragraph 3.9.1.4, the reactor vessel is further qualified to ensure against unstable crack growth under faulted conditions. Actuation of the emergency core cooling system (ECCS) following a loss-of-coolant accident produces relatively high thermal stresses in regions of the reactor vessel which come into contact with ECCS water. Primary consideration is given to these areas, including the reactor vessel beltline region and the reactor vessel primary coolant nozzle, to ensure the integrity of the reactor vessel under this severe postulated transient. The Westinghouse Owners Group evaluated TMI Action Item II.K.2.13, and the item is satisfied upon submittal of RT_{NDT} values which are below the pressurized thermal shock (PTS) rule screening values. Additionally, new

calculations were performed based on the requirements of Generic Letter 92-01 and the results are given in tables 5.3.3-2 and 5.3.3-3.

For the beltline region, significant developments have recently occurred in order to address PTS events. On the basis of recent deterministic and probabilistic studies, taking U.S. pressurized water reactor operating experience into account, the Nuclear Regulatory Commission staff concluded that conservatively calculated screening criterion values of RT_{NDT} less than 270° for plate material and axial welds, and less than 300° for circumferential welds, present an acceptably low risk of vessel failure from PTS events. These values were chosen as the screening criterion in the PTS rule for 10 CFR 50.34 (new plants) and 10 CFR 50.61 (operating plants).⁽²⁾ The conservative methods chosen by the NRC staff for the calculation of RT_{PTS} for the purpose of comparison with the screening criterion is presented in paragraph (b)(2) of 10 CFR 50.61. Details of the analysis method and the basis for the PTS rule can be found in SECY-82-465.⁽³⁾

The reactor vessel beltline materials are specified in subsection 5.3.1. The fluence of 4.76 x 10^{19} n/cm² which is the design basis fluence at the vessel inner radius, at 48 EFPY, at the peak location, was used for calculating all of the RT_{PTS} values. Based on the latest capsule data for each unit, the expected fluence at the vessel inner radius after 56.3 EFPY will be significantly less than the 4.76×10^{19} n/cm² design basis fluence³. RT_{PTS} is RT_{NDT}, the reference nilductility transition temperature as calculated by the method chosen by the NRC staff as presented in paragraph (b)(2) of 10 CFR 50.61, and the PTS rule. The PTS rule states that this method of calculating RT should be used in reporting values used to be compared to the above screening criterion set in the PTS rule. The screening criteria will not be exceeded using the method of calculation prescribed by the PTS rule for the vessel design lifetime. The material properties, initial RT_{NDT}, and end-of-life RT_{PTS} values are in tables 5.3.3-2 and 5.3.3-3. The materials identified in tables 5.3.3-2 and 5.3.3-3 are those materials that are exposed to high fluence levels at the beltline region of the reactor vessel and are, therefore, the subject of the PTS rule. These materials, therefore, are a subset of the materials identified in subsection 5.3.1.

The principles and procedures of linear elastic fracture mechanics (LEFM) are used to evaluate thermal effects in the regions of interest. The LEFM approach to the design against failure is basically a stress intensity consideration in which criteria are established for fracture instability in the presence of a crack. Consequently, a basic assumption employed in LEFM is that a crack or crack-like defect exists in the structure. The essence of the approach is to relate the stress field developed in the vicinity of the crack tip to the applied stress on the structure, the material properties, and the size of defect necessary to cause failure.

The elastic stress field at the crack tip in any cracked body can be described by a single parameter designated as the stress intensity factor, K. The magnitude of the stress intensity factor K is a function of the geometry of the body containing the crack, the size and location of the crack, and the magnitude and distribution of the stress.

The criterion for failure in the presence of a crack is that failure will occur whenever the stress intensity factor exceeds some critical value. For the opening mode of loading (stresses perpendicular to the major plane of the crack), the stress intensity factor is designated as K_I and the critical stress intensity factor is designated K_{IC} . Commonly called the fracture toughness, K_{IC} is an inherent material property which is a function of temperature. Any combination of applied load, structural configuration, crack geometry, and size which yields a stress intensity factor greater than K_{IC} for the material will result in crack instability.

The criterion of the applicability of LEFM is based on plasticity considerations at the postulated crack tip. Strict applicability (as defined by American Society of Testing Materials (ASTM)) of

^a Reactor vessel embrittlement is evaluated as a TLAA for license renewal (see subsection 19.4.1).

LEFM to large structures where plane strain conditions prevail requires that the plastic zone developed at the tip of the crack does not exceed approximately 2 percent of the crack depth. In the present analysis, the plastic zone at the tip of the postulated crack can reach 20 percent of the crack depth. However, LEFM has been successfully used to provide conservative brittle fracture prevention evaluations, even in cases where strict applicability of the theory is not permitted due to excessive plasticity. Recently, experimental results from the Heavy Section Steel Technology program intermediate pressure vessel tests have shown that LEFM can be applied conservatively as long as the pressure component of the stress does not exceed the yield strength of the material. The addition of the elastically calculated thermal stresses, which results in total stresses in excess of the yield strength, does not affect the conservatism of the results, provided that these thermal stresses are included in the evaluation of the stress intensity factors. Therefore, for faulted conditions analyses, LEFM is considered applicable for the evaluation of the vessel inlet nozzle and beltline region.

In addition, it has been well established that the crack propagation of existing flaws in a structure subjected to cyclic loading can be defined in terms of fracture mechanics parameters. Thus, the principles of LEFM are also applicable to fatigue growth of a postulated flaw at the vessel inlet nozzle and beltline region.

Additional details on this method of analysis of reactor vessels under severe thermal transients are given in reference 1.

5.3.3.7 Inservice Surveillance

The internal and external surfaces of the reactor vessel are accessible for periodic inspection. Visual and/or nondestructive techniques are used. During refueling, the vessel cladding is capable of being inspected in certain areas between the closure flange and the primary coolant inlet nozzles, and, if deemed necessary, the core barrel is capable of being removed, making the entire inside vessel surface accessible.

The closure head is examined to meet 10 CFR 50.55a requirements. Optical devices permit a selective inspection of the cladding, CRDM nozzles, and the gasket seating surface. The knuckle transition piece, which is the area of highest stress of the closure head, is accessible on the outer surface for visual inspection, dye penetrant or magnetic particle testing, and ultrasonic testing. The closure studs and nuts can be inspected periodically using visual, magnetic particle, and ultrasonic techniques.

The closure studs, nuts, washers, and the vessel flange seal surface, as well as the full-penetration welds in the following areas of the installed reactor vessel, are available for nondestructive examination:

- A. Vessel shell, from the inside and outside surfaces.
- B. Primary coolant nozzles, from the inside and outside surfaces. (a)
- C. Closure head, from the inside and outside surfaces; bottom head, from the inside and outside surfaces.
- D. Field welds between the reactor vessel nozzle safe ends and the main coolant piping, from the inside and outside surfaces.

The design considerations which have been incorporated into the system design to permit the above inspection are as follows:

-

⁽a) Only partial outside diameter coverage is provided.

- A. All reactor internals are completely removable. The tools and storage space required to permit these inspections are provided.
- B. The closure head is stored dry on the reactor operating deck during refueling to facilitate direct visual inspection.
- C. All reactor vessel studs, nuts, and washers can be removed to dry storage during refueling.
- D. Access is provided to the reactor vessel nozzle safe ends. The insulation covering the nozzle-to-pipe welds may be removed.

The reactor vessel presents access problems because of the radiation levels and remote underwater accessibility to this component. Because of these limitations on access to the reactor vessel, several steps have been incorporated into the design and manufacturing procedures in preparation for the periodic nondestructive tests which are required by the ASME inservice inspection code. These are as follows:

- A. Shop ultrasonic examinations are performed on all internally clad surfaces to an acceptance and repair standard to ensure an adequate cladding bond to allow later ultrasonic testing of the base metal from inside surface. The size of cladding bond defect allowed is 1/4 in. by 3/4 in. with the greater direction parallel to the weld in the region bounded by 2T (T = wall thickness) on both sides of each full-penetration pressure boundary weld. Unbounded areas exceeding 0.442 in.² (3/4-in. diameter) in all other regions are rejected.
- B. The design of the reactor vessel shell is an uncluttered cylindrical surface to permit future positioning of the test equipment without obstruction.
- C. The weld-deposited clad surface on both sides of the welds to be inspected is specifically prepared to ensure meaningful ultrasonic examinations.
- D. During fabrication, all full-penetration ferritic pressure boundary welds are ultrasonically examined in addition to code examinations.
- E. After the shop hydrostatic testing, all full-penetration ferritic pressure boundary welds (with the exception of the closure head welds), as well as the nozzles to safe end welds, are ultrasonically examined from both the inside and outside diameters in addition to ASME Code, Section III requirements. The closure head ferritic pressure boundary welds are examined from the outside diameter only.

The vessel design and construction enables inspection in accordance with the ASME Code, Section XI. The reactor vessel inservice inspection requirements are detailed in the VEGP inservice inspection program.

The Reactor Vessel Closure Head Stud Program is credited as a license renewal aging management program (see subsection 19.2.23).

5.3.3.8 References

- 1. Buchalet, C., Bamford, W. H., and Chirigos, J. N., "Method for Fracture Mechanics Analysis of Nuclear Reactor Vessels Under Severe Thermal Transients," <u>WCAP-8510</u>, December 1975.
- 2. PTS Rule, Federal Register Vol. 50, No. 141, July 23, 1985, 10 CFR 50.34.
- 3. NRC Policy Issue, "Pressurized Thermal Shock," SECY-82-465, November 23, 1982.

4.

EPRI NP 2712, "Feasibility of and Methodology for Thermal Annealing an Embrittled Reactor Vessel," November 1982.

TABLE 5.3.1-1 (SHEET 1 OF 2) REACTOR VESSEL QUALITY ASSURANCE PROGRAM

	RT ^(a)	<u>UT</u> ^(a)	PT ^(a)	MT ^(a)
Forgings Flanges Studs and nuts CRDM head adapter flange CRDM head adapter tube Instrumentation tube Main nozzles Nozzle safe ends		Yes Yes Yes Yes Yes Yes Yes	Yes Yes Yes	Yes Yes
Plates		Yes		Yes
Weldments CRDM head adapter to closure head connection Instrumentation tube to bottom head connection			Yes	
Main nozzle Cladding Nozzle to safe ends CRDM head adapter flange to CRDM head adapter	Yes Yes	Yes Yes Yes	Yes Yes Yes	Yes
tube All full-penetration ferritic pressure boundary welds accessible after	Yes		Yes	
hydrotest Full-penetration nonferritic pressure boundary welds accessible after hydrotest a. Nozzle to safe		Yes		Yes
ends b. CRDM head adapter flange to CRDM		Yes	Yes	
head adapter tube Seal ledge Head lift lugs			Yes	Yes Yes
Core pad welds			Yes	163

TABLE 5.3.1-1 (SHEET 2 OF 2)

- a. RT Radiographic.
 - UT Ultrasonic.
 - PT Dye penetrant.
 - MT Magnetic particle.

NOTE:

Base metal weld repairs as a result of UT, MT, RT, and/or PT indications are cleared by the same NDE technique/procedure by which the indications were found. The repairs meet all Section III requirements.

In addition, UT examination in accordance with the inprocess/posthydro UT requirements is performed on base metal repairs in the core region and base metal repairs in the inservice inspection zone (1/2 T).

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TABLE 5.3.1- $2^{(a)}$

VEGP UNIT 1 REACTOR VESSEL FRACTURE TOUGHNESS PROPERTIES

Use RT _{NDT} NMWD ^(b) (°F) (ff-lb)	15 88 88 85 20 132 0 0 132 132 132 132 132 132 132 132 132 132	
NDT (°F)	666000000000000000000000000000000000000	0, 1, 2, 4, 4, 8, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0,
P (%)	0.008 0.010 0.010 0.010 0.013 0.006 0.006 0.006 0.006 0.006 0.006 0.006 0.006 0.006 0.006 0.006 0.006	0.005 0.009 0.009 0.009 0.009
; <u>z</u> (%)	0.67 0.56 0.70 0.71 0.82 0.82 0.73 0.77 0.80 0.60 0.69 0.69	0.59 0.58 0.64 0.50 0.53
Cu (%)	0.00 0.00	0.05 0.05 0.06 0.13 0.10
Material Spec. No.	A533B Cl. 1 A508 Cl. 2 A508 Cl. 2	A533B Cl. 1 A533B Cl. 1 A533B Cl. 1 A533B Cl. 1 A533B Cl. 1 SAW
Code No.	B8807-1 B8808-1 B8809-1 B8809-2 B8809-3 B8809-3 B8810-2 B8810-4 B8810-4 B8804-1 B8804-1 B8804-1 B8804-1 B8805-1 B8805-1	B8606-1 B8606-2 B8606-3 B8813-1 B8812-1 G1.43
Component	Closure head dome Closure head torus(c) Closure head flange(c) Vessel flange(c) Inlet nozzle Inlet nozzle Inlet nozzle Outlet nozzle Outlet nozzle Outlet spelle Nozzle shell Intermediate shell(c) Intermediate shell(c) Intermediate shell(c) Intermediate shell(c) Intermediate shell(c)	Lower shell ^(c) Lower shell ^(c) Lower shell ^(c) Bottom head forus Bottom head dome Intermediate and lower ^(c)

a. This table is based on initial testing of the vessel materials at the time the surveillance capsule program was developed.

b. Normal to major working direction.

c. Denotes materials in reactor vessel closure flange and beltline region.

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TABLE 5.3.1-3^(a)

	Use NMWD ^(b) (ff-lb)	27 48 51 51 51 51 51 51 51 51 51 51
	RT _{NDT}	8 0 1 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
PERTIES	NDT (°F)	485666466666666666666666666666666666666
VESS PRO	P (<u>%)</u>	0.0008 0.0009 0.00009 0.0009 0.00009 0.00009 0.00009 0.00009 0.00009 0.00009 0.00009 0.00009 0.00009 0.00009 0.00009 0.00009 0.00009 0.00009 0.000009 0.00000000
TOUGH	i <u>N</u>	0.62 0.62 0.63 0.63 0.63 0.64 0.65 0.65 0.65 0.65 0.65 0.65 0.65 0.65
RACTURE	Cn (%)	0.07 0.07 0.09 0.09 0.09 0.00 0.00 0.05 0.05 0.05 0.05 0.05 0.05 0.05 0.06 0.05 0.06 0.06 0.06 0.06 0.06 0.07 0.08
CTOR VESSEL FI	Material Spec. No.	A533B CI. 1 A553B CI. 1 A508 CI. 2 A533B CI. 1
VEGP UNIT 2 REACTOR VESSEL FRACTURE TOUGHNESS PROPERTIES	Code No.	R9-1 R10-1 R7-1 R1-1 B9806-1 B9806-2 R6-3 R6-3 R6-3 R6-3 R6-3 R3-1 R3-2 R4-2 R4-2 R4-2 R4-2 R4-1 R3-1 R1-1 G1.60
	Component	Closure head dome Closure head flange(c) Closure head flange(c) Vessel flange(c) Inlet nozzle Inlet nozzle Outlet nozzle Outlet nozzle Outlet nozzle Outlet nozzle Outlet nozzle Intermediate shell Nozzle shell Nozzle shell Intermediate shell(c) Lower shell(c) Lower shell(c) Lower shell(c) Lower shell(c) Lower shell(c) Softom head dome Intermediate and lower(c) shell vertical weld seams and girth Intermediate and lower(c) shell vertical weld

a. This table is based on initial testing of the vessel materials at the time the surveillance capsule program was developed.

b. Normal to major working direction.

c. Denotes materials in reactor vessel closure flange and bettine region.

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TABLE 5.3.1-4

VEGP UNIT 1 REACTOR VESSEL CLOSURE HEAD BOLTING MATERIAL PROPERTIES

	Lateral Expansion (mils)	31, 30, 28 28, 32, 27 29, 36, 30 26, 26, 29 30, 34 27, 30, 31 27, 30, 31 27, 30, 31 27, 30, 31 27, 30, 31 27, 30, 31 27, 27, 27 27, 30, 31 28, 29, 30 27, 27, 27 27, 30, 31 28, 27, 30 31, 29, 30 31, 29, 27 31, 29, 29		29, 30, 27 30, 29, 29 28, 27, 28 32, 31, 34
	Energy at 10°F (ft-lb)	53, 54, 52 50, 51, 50 50, 51, 50 50, 51, 50 50, 51, 50 50, 51, 50 50, 52 50, 53 50, 54 50, 54 50, 54 50, 54 51, 50 51, 50		49, 52, 49 47, 49, 47 48, 48, 48 51, 51, 52
	Reduction in Area (%)	55.5 56.0 56.5 56.5 56.5 56.5 57.7 57.8 57.8 57.8 57.9 59.8 59.8 59.8 59.8		55.7 54.9 54.9 5.5
spr	Elongation (%)	77.0 77.0 77.0 77.0 76.5 76.5 76.5 76.0 77.0 76.0 76.0 76.0 76.0 76.0 76.0	Washers	17.0 17.0 16.0 17.5
Closure Head Studs	Ultimate Tensile Strength (ksi)_	162.5 163.0 162.5 162.0 164.0 165.0 165.0 156.0 153.0 153.0 153.0	Closure Head Nut and Washers	159.5 155.0 162.0 156.0
	0.2% Yield Strength (ksi)	4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	Ö	146.0 142.0 150.5 143.0
	Bar No.	166 166-1 167-1 170-1 172-1 172-1 197-1 201-1 207-1 212-1		75 75-1 78 78-1
	Material Spec. No.	SA540, B24 SA540, B24		SA540, B23 SA540, B23 SA540, B23 SA540, B23
	Heat No.	82029 82029 82029 82029 82029 82029 82029 82029 82552 82552 82552 82552 82552 82552 82552		19632 19632 19632 19632

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TABLE 5.3.1-5

VEGP UNIT 2 REACTOR VESSEL CLOSURE HEAD BOLTING MATERIAL PROPERTIES

	Lateral Expansion (mils)	31, 31, 33 30, 31, 33	26, 28, 27 31, 33, 30	28, 26, 29	27, 29, 26	26, 25, 29	27, 29, 29	28, 28, 30	29, 29, 27	27, 28, 29	32, 30, 30		3,	32,	3,	3,	33,	34,	36, 34, 32	30,
	Energy at 10°F (ft-lb)	50, 50, 51 48, 50, 51	47, 48, 47 50, 52, 50	47, 46, 48	47, 47, 46	47, 46, 48	47, 46, 48	48, 48, 49	47, 47, 47	48, 47, 48	50, 49, 49		51,	55,	53,	53,	54,	57,	55, 55, 54	53,
	Reduction in Area (%)	52.9 51.8	54.7 51.9	55.2	53.0	51.9	51.4	52.5	20.0	50.8	52.5		52.5	57.3	57.2	56.2	54.7	56.5	57.3	57.8
tuds	Elongation (%)	16.5 5.5	17.5 16.5	17.0	16.5	16.0	16.5	16.5	16.0	16.5	16.5	d Washers	17.0	18.5	18.5	18.0	18.0	18.0	17.0	17.5
Closure Head Studs	Ultimate Tensile Strength (ksi)_	159.0 160.0	164.0 162.0	163.5	163.0	162.5	165.0	163.0	157.0	158.5	155.0	Closure Head Nuts and Washers	163.0	158.5	161.5	162.0	161.0	160.0	157.0	158.5
	0.2% Yield Strength (ksi)	146.0 146.0	151.5 150.0	151.5	152.8	150.5	152.5	150.0	145.0	147.0	142.5	O	148.2	145.2	148.0	148.0	146.5	146.2	143.5	145.0
	Bar No.	265 265-1	271 271-1	273	273-1	278	278-1	283	283-1	286	286-1		105	105-1	110	110-1	113	113-1	117	117-1
	Material Spec. No.	SA540, B24 SA540, B24	SA540, B24 SA540, B24	SA540, B24		SA540, B24														
	Heat No.	83090 83090	83090 83090	83090	83090	83090	83090	83090	83090	83090	83090		83294	83294	83294	83294	83294	83294	83294	83294

TABLE 5.3.1-6

DELETED

TABLE 5.3.1-7

[HISTORICAL]

REACTOR VESSEL MATERIAL SURVEILLANCE PROGRAM

VEGP Unit 1

Capsules U, V, W, X, Y, and Z

<u>Material</u>	<u>Charpy</u>	<u>Tensile</u>	<u>1/2T-CT</u>
Plate B8805-3 (long.) Plate B8805-3 (trans.)	15 15	3 3	4 4
Weld metal (G-1.43)	15	3	4
HAZ	15	-	-

VEGP Unit 2

Capsules U, V, W, X, Y, and Z

<u>Material</u>	<u>Charpy</u>	<u>Tensile</u>	<u>1/2T-CT</u>
Plate B8628-1 (long.)	15 15	3 3	4
Plate B8628-1 (trans.)	15	3	4
Weld metal (E-3.23)	15	3	4
HAZ	15	-	-

TABLE 5.3.1-8

REACTOR VESSEL MATERIAL SURVEILLANCE PROGRAM WITHDRAWAL SCHEDULE (UNIT 1)

Capsule <u>Number</u>	Vessel Location	Lead <u>Factor^(a)</u>	Withdrawal Time <u>EFPY^(b)</u>	Approximate Capsule Fluence (n/cm², E > 1.0 MeV) (a)
U	58.5°	4.15	1.14	$3.34 \times 10^{18} (c)$
Υ	241°	3.99	4.85	1.16 x 10 ^{19 (c)}
V	61°	3.98	8.78	$1.97 \times 10^{19 (c) (d)}$
X	238.5°	4.21	14.33	$3.53 \times 10^{19 (c) (e)}$
W	121.5°	4.17	Standby	(f)
Z	301.5°	4.17	Standby	(f)

Updated in Capsule X dosimetry analysis.

b. Effective Full Power Years (EFPY) from plant startup.

c. Plant-specific evaluation.

d. This capsule was withdrawn at approximately the current end-of-license (36 EFPY) peak fluence.

e. This capsule was withdrawn at approximately (60 EFPY) peak fluence.

f. To be withdrawn at a fluence that is not less than once nor greater than twice the peak EOL fluence for an additional 20-year license renewal term to 80 years. Since the lead factor for both capsule W and Z are the same, either one may be withdrawn for 80 years license renewal.

TABLE 5.3.1-9

REACTOR VESSEL MATERIAL SURVEILLANCE PROGRAM WITHDRAWAL SCHEDULE (UNIT 2)

Capsule <u>Number</u>	Vessel Location	Lead <u>Factor^(a)</u>	Withdrawal Time <u>EFPY^(b)</u>	Approximate Capsule Fluence (n/cm², E > 1.0 MeV) (a)
U	58.5°	4.10	1.20	$3.56 \times 10^{18} (c)$
Υ	241°	3.95	4.98	1.12 x 10 ^{19 (c)}
X	238.5°	4.25	7.78	1.78 x 10 ^{19 (c)}
W	121.5°	4.14	13.29	$2.98 \times 10^{19 (c) (d)}$
Z	301.5°	4.15	18.48	4.16 x 10 ^{19 (c)}
V	61°	3.84	18.48 ^(e)	

a. Updated in Capsule W dosimetry analysis.

b. Effective Full Power Years (EFPY) from plant startup.

c. Plant-specific evaluation.

d. This capsule was withdrawn at a fluence not less than once nor greater than twice the peak EOL fluence for a standard license term of 40 years (36 EFPY). In addition, this capsule was withdrawn at a fluence not less than once nor greater than twice the peak EOL fluence for an additional 20-year license renewal term to 60 years (54 EFPY).

e. Capsule V has been removed from the reactor vessel and placed in the spent fuel pool. No testing or analysis has been performed on this capsule. Reinsertion of this capsule may be considered in the future.

TABLE 5.3.2-1

DELETED

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VOGTLE UNIT 1 REACTOR VESSEL CORE BELTLINE REGION TOUGHNESS PROPERTIES

TABLE 5.3.2-2 (SHEET 1 OF 2)

Intermediate Shell Course

	Shear (%)	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0		
	Lat. Exp. (mils)	1 0 7 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2		
Plate B8805-3	Energy <u>(ft lb)</u>	10 10 10 10 10 10 10 10 10 10 10 10 10 1		ا∘ل
	Тетр. (°F)	0 4 4 4 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	T _{NDT} = -20°F	RT _{NDT} = +30°F
	Shear (%)_	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0		
	Lat. Exp. (mils)_	otus 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8		
Plate B8805-2	Energy (ft lb)	24 - 7 8 8 8 8 9 4 4 5 4 5 4 5 6 9 8 8 8 9 9 6 6 6 6 6 6 6 6 6 6 6 6 6		LL.
	Temp. (°F)	04 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	$T_{NDT} = -10^{\circ}F$	$RT_{NDT} = +20^{\circ}F$
	Shear (%)	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0		
	Lat. Exp (mils)	7 + 1 + 2 + 2 + 2 + 2 + 8 + 8 + 8 + 8 + 8 + 8		
Plate B8805-1	Energy <u>(ft lb)</u>	121 81888888 8187 8188888 81888888888888		
A.	Temp. (°F)_	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	$T_{NDT} = 0^{\circ}F$	$RT_{NDT} = 0^{\circ}F$

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TABLE 5.3.2-2 (SHEET 2 OF 2)

Lower Shell Course

	Shear (%)	0 0 0 0 1 0 1 0 0 0 0 0 0 0 0 0 0 0 0 0	
	Lat. Exp. (mils)	6 4 4 4 1 1 1 0 8 8 8 8 8 8 8 8 9 5 8 8 8 8 8 8 8 8 8 8	
Plate B8606-3	Energy (ft lb)	6 7 8 8 8 7 8 9 9 8 4 9 4 9 4 9 9 8 9 9 9 9 9 9 9 9	
	Temp. (°F)	-40 -40 -40 -40 -40 -40 -40 -40	
	Shear (%)	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
	Lat. Exp. (mils)	7	
Plate B8606-2	Energy (ft lb)	8 2 5 5 7 7 7 7 9 8 9 9 9 9 9 9 9 9 9 9 9 9 9 9	
	Temp. (°F)	40 40 40 10 10 10 40 40 40 40 70 70 80 80 80 80 100 100 160 212 212 212 212 212 212 212 212 212 21	
	Shear (%)	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
	Lat. Exp (mils)	7 9 9 7 7 7 8 8 8 7 7 7 8 8 8 7 7 7 8 8 8 7 7 7 8 8 7 7 8 8 7 7 8 8 7 7 8 8 7 8 9 7 8 9 7 8 9 9 9 9	
Plate B8606-1	Energy (ft lb)	98888777777777777777777777777777777777	
₫.	Temp. (°F)	40 40 40 10 10 40 40 40 40 70 70 70 70 80 80 80 80 100 100 100 100 100 100 10	

TABLE 5.3.2-3

VEGP UNIT 1 REACTOR VESSEL CORE BELTLINE REGION WELD METAL TOUGHNESS PROPERTIES

Intermediate and Lower Shell Long, and Girth Weld Seams Weld Code No. G1.43

Temp.	Energy	Lat. Exp.	Shear
(°F)	(ft lb)	<u>(mils)</u>	(%)
-80	6	2	0
-80	5	1	0
-80	6	1	0
-40	65	44	35
-40	10	3	0
-40	8	2	0
10	124	77	80
10	114	66	70
10	96	60	60
60	120	71	70
60	118	70	80
60	119	76	70
100	124	80	80
100	127	78	80
100	123	80	80
160	127	78	100
160	136	82	100
160	140	84	100

 $T_{NDT} = -80$ °F

 $RT_{NDT} = -80$ °F

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TABLE 5.3.2-4 (SHEET 1 OF 2)
VEGP UNIT 2 REACTOR VESSEL CORE BELTLINE REGION TOUGHNESS PROPERTIES

Intermediate Shell Course

	Shear (%)	c	>	0	0	2	2	2	15	20	35	30	30	30	40	35	35	20	20	80	100	100	100		
	Lat. Exp. (mils)	¢	0	വ	4	16	15	18	30	98	43	35	88	41	51	46	4	51	20	26	89	99	61		
Plate R4-3	Energy <u>(ff lb)</u>	c	ກ	7	7	19	18	19	37	40	55	42	44	48	29	52	20	20	29	74	98	89	78	r _{NDT} = 0°F	₹T _{NDT} = 30°F
	Temp. (°F)	,	04-	40	40	10	10	10	09	09	09	80	80	80	06	06	06	100	100	100	160	160	160	F	RT
	Shear (%)	c	>	0	0	10	15	20	15	20	20	40	40	25	40	35	30	80	80	92	100	100	100		
	Lat. Exp. (mils)	•	4	4	2	17	27	31	28	34	35	49	20	34	53	47	39	29	64	65	69	20	74		
Plate R4-2	Energy <u>(ft lb)</u>	¢	0	7	œ	23	33	38	36	40	43	63	99	48	99	29	51	88	85	96	103	66	110	0∘F	10°F
	Temp. (°F)	Ç	-40	40	40	10	10	10	20	20	20	09	09	09	20	20	20	100	100	100	160	160	160	T _{NDT} = -10°F	RT _{NDT} = 10°F
	Shear (%)	c	>	0	0	15	10	15	25	25	20	35	35	30	40	20	40	80	06	06	100	100	100		
	Lat. Exp (mils)	Ç	0	വ	7	24	19	24	33	34	31	42	43	36	47	54	43	65	29	20	72	69	29		
Plate R4-1	Energy (ft lb)	7	_	10	12	30	27	33	43	45	40	52	51	46	61	72	54	80	98	91	26	92	93	ĻĿ	پ
	Temp. (°F)	,	04	40	-40	10	10	10	40	40	40	09	09	09	20	20	20	100	100	100	160	160	160	T _{NDT} = -20°F	RT _{NDT} = 10°F

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TABLE 5.3.2-4 (SHEET 2 OF 2)

Lower Shell Course

	Shear (%)	0	0	0	10	10	10	30	30	30	40	30	35	20	20	20	92	92	92	100	100	100	100	100	100		
	Lat. Exp. (mils)	4	က	4	12	7		20	18	17	45	35	36	20	49	45	09	29	28	09	28	22	09	65	89		
Plate B8628-1	Energy (ft lb)	17	6	7	15	15	15	26	25	22	55	43	48	61	61	28	92	20	89	92	20	20	9/	82	96	_{NDT} = -20°F	(T _{NDT} = 50°F
	Temp. (°F)	4	-40	4	0	0	0	40	40	40	100	100	100	110	110	110	160	160	160	212	212	212	275	275	275	F	R
	Shear (%)	2	0	0	20	20	20	25	25	25	40	09	09	65	65	65	92	92	92	100	100	100					
	Lat. Exp. (mils)	7	2	4	18	16	16	22	23	25	39	46	46	20	51	52	64	92	99	61	64	89					
Plate R8-1	Energy (ft lb)	4	12	7	22	22	20	33	34	35	49	62	62	29	89	69	82	98	06	83	98	93				J∘L	= 40°F
	Temp. (°F)	-40	40	40	0	0	0	40	40	40	06	06	06	100	100	100	160	160	160	212	212	212				$T_{NDT} = -20^{\circ}F$	$RT_{NDT} = 4$
	Shear (%)	0	0	0	10	10	10	30	30	30	20	20	20	20	20	20	06	06	06	100	100	100					
	Lat. Exp (mils)	7	80	2	10	တ	6	25	20	23	38	35	42	41	47	49	09	64	09	99	92	64					
Plate B8825-1	Energy (ft.lb)	17	11	10	16	14	14	37	28	32	51	49	26	55	09	61	9/	79	77	98	82	80				Ŀ	9°F
	Temp. (°F)	40	-40	-40	0	0	0	40	40	40	06	06	06	100	100	100	160	160	160	212	212	212				$T_{NDT} = -20^{\circ}F$	$RT_{NDT} = 40^{\circ}F$

TABLE 5.3.2-5 (SHEET 1 OF 2)

VEGP UNIT 2 REACTOR VESSEL CORE BELTLINE REGION WELD METAL TOUGHNESS PROPERTIES

Intermediate to Lower Shell Girth Weld Seam

	Weld Code	No. E3.23	
Temp.	Energy	Lat. Exp.	Shear
(°F)	(ft lb)	<u>(mils)</u>	(%)
			
-80	6	4	0
-80	17	13	0
-80	12	8	0
-40	43	30	20
-40	39	28	15
-40	12	6	0
10	53	33	25
10	45	39	20
10	52	30	25
20	28	19	10
20	68	51	50
20	64	50	50
30	58	45	30
30	59	46	35
30	60	46	35
60	68	54	70
60	71	57	70
60	62	44	60
100	92	72	100
100	90	74	80
100	87	68	90
160	88	69	100
160	94	70	100
160	89	68	100

 $T_{NDT} = 50$ °F

 $RT_{NDT} = -30$ °F

TABLE 5.3.2-5 (SHEET 2 OF 2)

Intermediate and Lower Shell Long Weld Seams

Weld Code No. 1.60

Temp. (°F)	Energy (ft lb)	Lat. Exp. (mils)	Shear (%)
-30	15	10	0
-30	23	15	5 5
-30	24	17	5
-10	45	31	20
-10	57	41	30
-10	51	37	25
10	120	82	95
10	105	70	70
10	110	76	80
30	118	70	70
30	111	64	60
30	125	76	80
50	142	86	100
50	153	90	100
50	145	87	100
100	156	88	100
100	159	83	100
100	141	79	100

 $T_{NDT} = 10$ °F

 $RT_{NDT} = -10$ °F

TABLE 5.3.3-1

REACTOR VESSEL DESIGN PARAMETERS

Design/operating pressure (psig)	2485/2235
Design temperature (°F)	650
Overall height of vessel and closure head, bottom head outside diameter to top of control rod mechanism adapter (ft-in.)	43-10
Thickness of reactor pressure vessel head insulation, minimum (in.)	3
Number of reactor closure head studs	54
Diameter of reactor closure head/studs, minimum shank (in.)	6 13/16
Outside diameter of flange (in.)	205
Inside diameter of flange (in.)	167
Outside diameter at shell (in.)	190 1/2
Inside diameter at shell (in.)	173
Inlet nozzle inside diameter (in.)	27 1/2
Outlet nozzle inside diameter (in.)	29
Clad thickness, minimum (in.)	1/8
Lower head thickness, minimum (in.)	5 3/8
Vessel beltline thickness, minimum (in.)	8 5/8
Closure head thickness (in.)	7
Nominal water volume (ft ³)	3700

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UNIT 1 REACTOR VESSEL VALUES FOR ANALYSIS OF POTENTIAL PRESSURIZED THERMAL SHOCK EVENTS^(a) **TABLE 5.3.3-2**

			Initial	10 CFR 50.61 I RI _{P1}	Predicted 's (°F)		Regulato Predict	Regulatory Guide 1.99 Predicted USE (ft-lb)
<u>Material</u>	Cu wt-%	Ni wt-%	T	36 EFPY	57 <u>EFPY</u>	Initial USE <u>(ff-lb)</u>	36 EFPY	57 EFPY
Intermediate Shell Plate, B8805-1	0.083	0.597	0	86	104	06	72	69
Intermediate Shell Plate, B8805-2	0.083	0.61	20	118	124	100	80	77
Intermediate Shell Plate, B8805-3 ^(b)	0.062	0.598	30	110 121 ^(d)	115 126 ^(d)	107	86 ^(e)	82 ^(e)
Lower Shell Plate, B8606-1	0.053	0.593		94	26	116	93	88
Lower Shell Plate, B8606-2	0.057	09.0		26	101	113	06	87
Lower Shell Plate, B8606-3	0.067	0.623	10	92	100	118	94	91
Intermediate Shell Longitudinal Weld Seams 101-124 A, B, & C $^{\rm (c)}$	0.042	0.102		3 11 -24 ^(d) -21 ^(d)	11 -21 ^(d)	134	107 ^(e)	103 ^(e)
Lower Shell Longitudinal Weld Seams 101-142 A, B, & C ^(c)	0.042	0.102	-80	3 -24 ^(d)	11 -21 ^(d)	134	107 ^(e)	103 ^(e)
Intermediate to Lower Shell Girth Weld 101-171 ^(c)	0.042	0.102	-80	3 -24 ^(d)	11 -21 ^(d)	134	107 ^(e)	103 ^(e)

NOTES:

- RT_{PTS} values are based on the peak fluences at the vessel inner radius of 2.155 E19 (for 36 EFPY) and 3.485 E19 (for 57 EFPY). USE was predicted using the 1/4T fluence values based on the peak fluence at the vessel inner radius. The vessel wall thickness is 8.625 inches at the beltline region. Copper and nickel values for all materials are the latest Best Estimate values as of April 2005. æ.
 - Limiting vessel material for Pressurized Thermal Shock event.
 - ю с с р
- All of the core region welds were fabricated from wire heat 83653, linde 0091 flux, lot # 3536.

 Determined using 10 CFR 50.61 with credible surveillance capsule data for the welds and noncredible surveillance capsule data for the plate.

 Conservatively determined using Position 1.2 (without surveillance capsule data) of Regulatory Guide 1.99, Revision 2; however, surveillance data were available.

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UNIT 2 REACTOR VESSEL VALUES FOR ANALYSIS OF POTENTIAL PRESSURIZED THERMAL SHOCK EVENTS^(a)

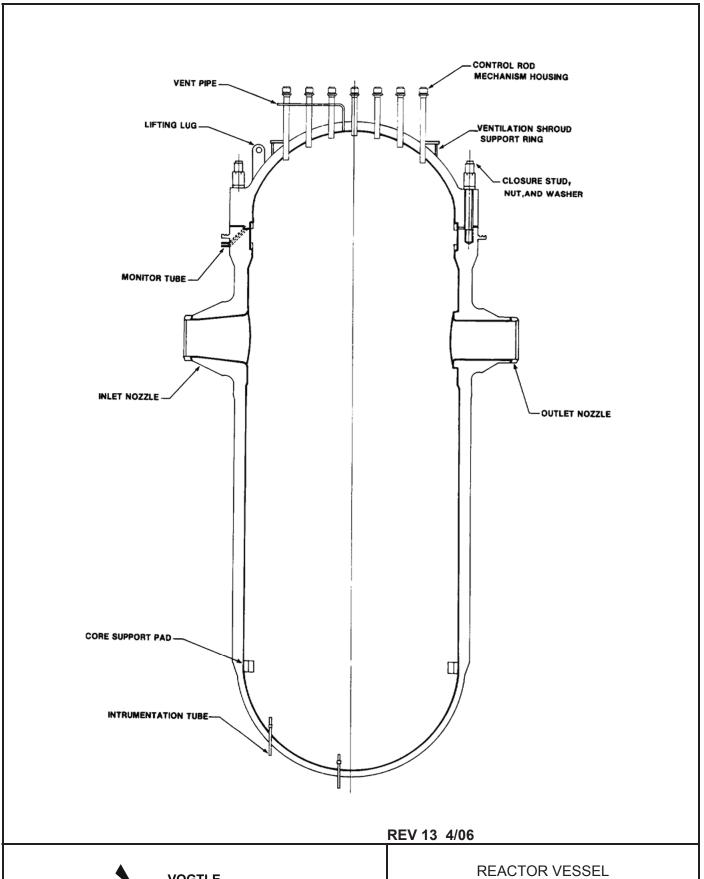
TABLE 5.3.3-3

				FR 50.61 Pr	edicted		Regulatory Gui	de 1.99
Material	Cu wt-%	Ni wt-%	_ T0	RT _{PTS} 36 EFPY	(°F) 57 EFPY	Initial USE (ft-lb)	Predicted USE (ft-lb) 36 57 EFPY EFPY	E (ff-lb) 57 EFPY
Intermediate Shell Plate, R4-1	0.07	0.63	10	95.9	101.0		92	74
Intermediate Shell Plate, R4-2	90.0	0.61		87.7	91.9	104	83	81
Intermediate Shell Plate, R4-3	0.05	09.0	30	100.6	104.1		29	99
Lower Shell Plate, B8825-1	90.0	0.62		117.7	121.9	83	99	65
Lower Shell Plate, R8-1 ^(b)	0.07	0.63	40	125.9	131.0	87	70	89
Lower Shell Plate, B8628-1	0.05	0.59	50	120.6 91.1 ^(d)	124.1 93.4 ^(d)	85	79 ^(e)	79 ^(e)
Intermediate Shell Longitudinal Weld Seams 101-124 A, B, & C ^(c)	0.05	0.15	-10	92.2 40.8 ^(d)	92.2 102.1 40.8 ^(d) 45.6 ^(d)	152	140 ^(e)	140 ^(e)
Lower Shell Longitudinal Weld Seams 101-142 A, B, & C ^(c)	0.05	0.15	-10	92.2 40.8 ^(d)	102.1 45.6 ^(d)	152	140 ^(e)	140 ^(e)
Intermediate to Lower Shell Girth Weld ^(c)	0.05	0.15	-30	72.2 20.8 ^(d)		06	83 ^(e)	83 _(e)

NOTES:

- RT_{PTS} values are based on the peak fluence at the vessel inner radius of 1.93 E19 (for 36 EFPY) and 3.06 E19 (for 57 EFPY). USE was predicted using the 1/4T fluence values based on the peak fluence at the vessel inner radius. The vessel wall thickness is 8.625 inches at the beltline region. Copper and nickel values for all materials are the latest Best Estimate values as of April 2005. æ.
 - Limiting vessel material for Pressurized Thermal Shock event.
- The longitudinal welds were fabricated from wire heat 87005 linde 0091 flux, lot 0145. The girth weld was fabricated from weld wire heat 87005, linde 124 flux, lot 1061. Determined using 10 CFR 50.61 with credible surveillance capsule data. Determined using Position 2.2 (with credible surveillance capsule data) of Regulatory Guide 1.99, Revision 2.

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FIGURE 5.3.3-1

5.4 COMPONENT AND SUBSYSTEM DESIGN

5.4.1 REACTOR COOLANT PUMP ASSEMBLY

5.4.1.1 <u>Design Bases</u>

The reactor coolant pump assembly ensures an adequate core cooling flowrate for sufficient heat transfer to maintain a departure from nucleate boiling ratio greater than the design basis limit within the parameters of operation. The required net positive suction head (NPSH) is by conservative pump design always less than that available by system design and operation.

Sufficient pump assembly rotation inertia is provided by a motor flywheel, motor rotor, and pump rotating parts which provide adequate flow during coastdown conditions. This forced flow following an assumed loss of offsite electrical power and the subsequent natural circulation effect provides the core with adequate cooling.

The reactor coolant pump is shown in figure 5.4.1-1. The reactor coolant pump design parameters are given in table 5.4.1-1.

The automatic trip of the reactor coolant pumps discussed in TMI Action Item II.K.3.5 has not been implemented. Generic studies⁽³⁾⁽⁴⁾ performed in response to Nuclear Regulatory Commission Generic Letters 83-10c and d⁽⁵⁾ have demonstrated the acceptability of manual tripping of the reactor coolant pumps following accident events postulated to occur at VEGP.

The auxiliary component cooling water (ACCW) pump motors are accessible to the emergency diesel generators as described in section 8.3 and meet the intent of TMI Action Item II.K.3.25.

5.4.1.2 Pump Assembly Description

5.4.1.2.1 Design Description

The reactor coolant pump is a vertical, single-stage, controlled leakage, centrifugal pump designed to pump large volumes of reactor coolant at high temperatures and pressures.

The pump assembly consists of three major sections. They are the hydraulics, the seals, and the motor.

- A. The hydraulic section consists of the casing, impeller, turning vane diffuser, and diffuser adapter.
- B. The shaft seal section consists of the No. 1 controlled leakage film-riding face seal, No. 2 and No. 3 rubbing face seals, and a shutdown seal assembly (SDS). The seals are contained within the thermal barrier heat exchanger assembly and seal housing. The SDS is housed within the No. 1 seal area and is a passive device actuated by high temperature resulting from a loss of seal injection and ACCW cooling to the thermal barrier heat exchanger.

C. The motor section consists of a drip-proof squirrel cage induction motor with a vertical solid shaft, an oil-lubricated, double-acting Kingsbury type thrust bearing, upper and lower oil-lubricated radial guide bearings, and a flywheel.

Additional components of the pump are the shaft, pump radial bearing, thermal barrier heat exchanger assembly, coupling, spool piece, and motor stand.

In the event of a loss of seal injection and ACCW flow to the thermal barrier heat exchanger, the SDS will actuate only when the No. 1 seal temperature reaches 250°F to 300°F. SDS actuation limits leakage from the RCS through the RCP seal package. Leakage is limited when the SDS thermal actuator retracts due to intrusion of hot reactor coolant water into the seal area, which causes the SDS piston and polymer rings to constrict around the shaft.

5.4.1.2.2 Description of Operation

The reactor coolant enters the suction nozzle, is pumped by the impeller through the diffuser, and exits through the discharge nozzle. The diffuser adapter limits the leakage of reactor coolant back to the suction.

Seal injection flow, under slightly higher pressure than the reactor coolant, enters the pump through a connection on the thermal barrier flange and is directed into the plenum between the thermal barrier housing and the shaft. The flow splits, with the major portion flowing down the shaft through the radial bearing and into the RCS. The remaining seal injection flow passes up the shaft through the seals.

The ACCW is provided to the thermal barrier heat exchanger. During normal operation, the thermal barrier limits the heat transfer from hot reactor coolant to the radial bearing and to the seals. In addition, if a loss of seal injection flow should occur, the thermal barrier heat exchanger cools the reactor coolant to an acceptable level before it enters the bearing and seal area.

The reactor coolant pump motor oil-lubricated bearings are of conventional design. The radial bearings are the segmented pad type, and the thrust bearing is a double-acting Kingsbury type.

Auxiliary component cooling water is supplied to the external upper bearing oil cooler and to the integral lower bearing oil cooler.

The oil spillage protection system is attached to the reactor coolant pump motor and is provided to contain and channel oil to a common collection point.

The motor is a drip-proof squirrel cage induction motor with Class F thermalastic epoxy insulation, fitted with external water/air coolers. The rotor and stator are of standard construction. Six resistance temperature detectors are embedded in the stator windings to sense stator temperature. A flywheel and an antireverse rotation device are located at the top of the motor.

The internal parts of the motor are cooled by air. Integral vanes on each end of the rotor draw air in through cooling slots in the motor frame. This air passes through the motor with particular emphasis on the stator end turns. It is then routed to the external water/air coolers, which are supplied with ACCW. Each motor has two such coolers, mounted diametrically opposed to each other. Coolers are sized to maintain optimum motor-operating temperature. The air is finally exhausted to the containment environment.

Each of the reactor coolant pump assemblies is equipped for continuous monitoring of reactor coolant pump shaft and frame vibration levels. Shaft vibration is measured by two relative shaft probes mounted on top of the pump seal housing; the probes are located 90° apart in the same

horizontal plane and mounted near the pump shaft. Frame vibration is measured by two velocity seismoprobes located 90° apart in the same horizontal plane and mounted at the top of the motor support stand. Proximeters and converters linearize the probe output, which is displayed on monitor meters in the control building, with alarm functions available in the control room. The monitor meters automatically indicate the highest output from the relative probes and seismoprobes; manual selection allows monitoring of individual probes. Indicator lights display caution and danger limits of vibration.

The spool piece, which is a removable shaft segment, is located between the motor coupling flange and the pump coupling flange; the spool piece allows removal of the pump seals with the motor in place. The pump internals, motor, and motor stand can be removed from the casing without disturbing the reactor coolant piping. The flywheel is available for inspection by removing the cover.

All parts of the pump in contact with the reactor coolant are austenitic stainless steel except for seals, bearings, and special parts.

5.4.1.2.3 Loss of Seal Injection

Should a loss of seal injection to the reactor coolant pumps occur, the pump radial bearing and seals are lubricated by reactor coolant flowing up through the pump. Under these conditions, the ACCW continues to provide flow to the thermal barrier heat exchanger; this heat exchanger, functioning in its backup capacity, cools the reactor coolant before it enters the pump radial bearing and the shaft seal area. The loss of seal injection flow may result in a temperature increase in the pump bearing area, a temperature increase in the seal area, and a resultant increase in the No. 1 seal leak rate; however, pump operation can be continued provided these parameters remain within the allowable limits. Under certain low seal leak-off conditions it is possible that seal temperatures may increase above allowable limits in 1 to 2 hours requiring shutdown of the pump.

5.4.1.2.4 Loss of Auxiliary Component Cooling Water

Should a loss of ACCW to the reactor coolant pumps occur, the chemical and volume control system continues to provide seal injection flow to the reactor coolant pumps; the seal injection flow is sufficient to prevent damage to the seals with a loss of thermal barrier cooling. However, the loss of ACCW to the motor bearing oil coolers will result in an increase in oil temperature and a corresponding rise in motor bearing metal temperature. It has been demonstrated by testing at the Westinghouse Electromechanical Division that the reactor coolant pumps will incur no damage as a result of an ACCW flow interruption of 10 min.

Safety-related transmitters will be provided to redundantly monitor ACCW flow for the upper and lower reactor coolant pump bearings, as well as to monitor ACCW flow for the reactor coolant pump thermal barriers. These transmitters will provide flow indication and actuate low-flow alarms in the control room.

Operating procedures are provided for a loss of ACCW and seal injection to the reactor coolant pumps and/or motors. Included in these operating procedures is the provision to trip the reactor if ACCW flow, as indicated by the instrumentation discussed above, is lost to the reactor coolant pump motors and cannot be restored within 10 min. The reactor coolant pumps will also be tripped following the reactor trip.

5.4.1.3 <u>Design Evaluation</u>

5.4.1.3.1 Pump Performance

The reactor coolant pumps are sized to deliver flow at rates which equal or exceed the required flowrates. Initial RCS tests confirm the total delivery capability. Thus, assurance of adequate forced circulation coolant flow is provided prior to initial plant operation.

The estimated performance characteristic is shown in figure 5.4.1-2. The knee, at about 25% design flow, introduces no operational restrictions, since the pumps only operate at a speed which corresponds to full flow.

The reactor coolant pump motor is tested, without mechanical damage, at overspeeds up to and including 125% of normal speed. The integrity of the flywheel during a loss-of-coolant accident (LOCA) is demonstrated in reference 1, which is undergoing generic review by the NRC.

The reactor trip system ensures that pump operation is within the assumptions used for loss-of-coolant flow analyses, which also ensures that adequate core cooling is provided to permit an orderly reduction in power if flow from a reactor coolant pump is lost during operation.

An extensive test program has been conducted for several years to develop the controlled leakage shaft seal for pressurized water reactor applications. Long-term tests were conducted on less than full-scale prototype seals as well as on full-size seals. Operating plants continue to demonstrate the satisfactory performance of the controlled leakage shaft seal pump design.

The support of the stationary member of the No. 1 seal (seal ring) is such as to allow deflections, both axial and tilting, while still maintaining its controlled gap relative to the seal runner. Even if all the graphite were removed from the pump bearing, the shaft could not deflect far enough to cause opening of the controlled leakage gap. The spring rate of the hydraulic forces associated with the maintenance of the gap is high enough to ensure that the ring follows the runner under very rapid shaft deflections.

Testing of pumps with the No. 1 seal entirely bypassed (full system pressure on the No. 2 seal) shows that relatively small leakage rates would be maintained for a period of time which is sufficient to secure the pump; even if the No. 1 seal fails entirely during normal operation, the No. 2 seal would maintain these small leakage rates if the proper action is taken by the operator. The plant operator is warned of No. 1 seal damage by an increase in No. 1 seal leakoff rate. Following warning of excessive seal leakage conditions, the plant operator should close the No. 1 seal leakoff line and secure the pump, as specified in the instruction manual.

Gross leakage from the pump does not occur if the proper operator action is taken subsequent to warning of excessive seal leakage conditions.

The effect of loss of offsite power on the pump itself is to cause a temporary stoppage in the supply of injection flow to the pump seals and also of the ACCW for seal and bearing cooling. The emergency diesel generators are started automatically due to loss of offsite electrical power, so that ACCW flow and seal injection flow are automatically restored.

5.4.1.3.2 Coastdown Capability

It is important to reactor protection that the reactor coolant continues to flow for a short time after reactor trip. In order to provide this flow following loss of offsite electrical power, each reactor coolant pump is provided with a flywheel. Thus, the rotating inertia of the pump, motor, and flywheel is employed during the coastdown period to continue the reactor coolant flow. The

coastdown flow transients are provided in the figures in section 15.3. The pump/motor system is designed for the safe shutdown earthquake at the site. Hence, it is concluded that the coastdown capability of the pumps is maintained even under the most adverse case of loss of offsite electrical power coincident with the safe shutdown earthquake. Core flow transients and figures are provided in subsections 15.3.1 and 15.4.4. An inadvertent actuation of the SDS on the rotating assembly will not have any measurable impact on the RCP coastdown.

5.4.1.3.3 Bearing Integrity

The design requirements for the reactor coolant pump bearings are primarily aimed at ensuring a long life with negligible wear, so as to give accurate alignment and smooth operation over long periods of time. The surface bearing stresses are held at very low values and even under the most severe seismic transients do not begin to approach loads that cannot be adequately carried for short periods of time.

Because there are no established criteria for short-time, stress-related failures in such bearings, it is not possible to make a meaningful quantification of such parameters as margins to failure, safety factors, etc. A qualitative analysis of the bearing design, embodying such considerations, gives assurance of the adequacy of the bearing to operate without failure.

Low oil levels in the motor lube oil sumps signal alarms in the control room. Each motor bearing contains embedded temperature detectors, so that initiation of failure, separate from loss of oil, is indicated and alarmed in the control room as a high bearing temperature. This requires pump shutdown. If these indications are ignored and the bearing proceeds to failure, the low melting point of Babbitt metal on the pad surfaces ensures that sudden seizure of the shaft will not occur. In this event, the motor continues to operate, since it has sufficient reserve capacity to drive the pump under such conditions. However, the high torque required to drive the pump will require high current, which leads to the motor being shutdown by the electrical protection systems.

5.4.1.3.4 Locked Rotor

It may be hypothesized that the pump impeller might severely rub on a stationary member and then seize. Analysis has shown that under such conditions, assuming instantaneous seizure of the impeller, the pump shaft fails in torsion just below the coupling to the motor, disengaging the flywheel and motor from the shaft. This constitutes a loss of coolant flow in the loop. Following such a postulated seizure, the motor continues to run without any overspeed, and the flywheel maintains its integrity, since it is still supported on a shaft with two bearings. Flow transients are provided in the figures in subsection 15.3.3 for the assumed locked rotor.

There are no other credible sources of shaft seizure other than impeller rubs. A sudden seizure of the pump bearing is precluded by graphite in the bearing. Any seizure in the seals results in a shearing of the antirotation pin in the seal ring. The motor has adequate power to continue pump operation even after the above occurrences. Indications of pump malfunction in these conditions are initially given by high-temperature signals from the bearing water temperature detector, by excessive No. 1 seal leakoff indications, and by offscale No. 1 seal leakoff indications. Following these signals, pump vibration levels are checked. Excessive vibration indicates mechanical trouble, and the pump is shut down for investigation. Note an inadvertent actuation of the SDS on a rotating assembly will not prevent sufficient cooling flow to the reactor core.

5.4.1.3.5 Critical Speed

The reactor coolant pump shaft is designed so that its operating speed is below its first critical speed. This shaft design, even under the most severe postulated transient, gives low values of actual stress.

5.4.1.3.6 Missile Generation

Precautionary measures taken to preclude missile formation from reactor coolant pump components ensure that the pumps will not produce missiles under any anticipated accident condition

Appropriate components of the reactor coolant pump have been analyzed for missile generation. Any fragments of the motor rotor would be contained by the heavy stator frame. The same conclusion applies to the pump impeller because the small fragments that might be ejected would be contained by the heavy casing. Further discussion and analysis of missile generation are contained in reference 1.

5.4.1.3.7 Pump Cavitation

The minimum NPSH required by the reactor coolant pump at best estimate flow is approximately a 300-ft head (approximately 133 psi). In order for the controlled leakage seal to operate correctly, it is necessary to require a minimum differential pressure of approximately 200 psi across the No. 1 seal. This corresponds to a primary loop pressure at which the minimum NPSH is exceeded, and no limitation on pump operation occurs from this source.

5.4.1.3.8 Pump Overspeed Considerations

For turbine trips actuated by either the reactor trip system or the turbine protection system, the generator and reactor coolant pumps remain connected to the external network for 30 s to prevent any pump overspeed condition.

An electrical fault requiring immediate trip of the generator (with resulting turbine trip) could result in an overspeed condition. However, the turbine control system and the turbine intercept valves limit the overspeed to less than 120%. As additional backup, the turbine protection system utilizes redundant overspeed protection with trip manifold assemblies using primary and emergency overspeed trips set at 110% and 110.5% (of turbine speed). In case a generator trip deenergizes the pump buses, the reactor coolant pump motors are transferred from the unit auxiliary transformers to the reserve auxiliary transformers within 6 to 10 cycles. Further discussion of pump overspeed considerations is contained in reference 1.

5.4.1.3.9 Antireverse Rotation Device

Each of the reactor coolant pumps is provided with an antireverse rotation device in the motor. This antireverse mechanism consists of pawls mounted on the outside diameter of the flywheel, a serrated ratchet plate mounted on the motor frame, a spring return for the ratchet plate, and two shock absorbers.

At an approximate forward speed of 70 rpm, the pawls drop and bounce across the ratchet plate; as the motor continues to slow, the pawls drag across the ratchet plate. After the motor

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has slowed and come to a stop, the dropped pawls engage the ratchet plate, and as the motor tends to rotate in the opposite direction, the ratchet plate also rotates until it is stopped by the shock absorbers. The rotor remains in this position until the motor is energized again. When the motor is started, the ratchet plate is returned to its original position by the spring return.

As the motor begins to rotate, the pawls drag over the ratchet plate. When the motor reaches sufficient speed, the pawls are bounced into an elevated position and are held in that position by friction resulting from centrifugal forces acting upon the pawls. While the motor is running at speed, there is no contact between the pawls and ratchet plate.

Considerable plant experience with the design of the antireverse rotation device has shown high reliability of operation.

5.4.1.3.10 Shaft Seal Leakage

During normal operation, leakage along the reactor coolant pump shaft is controlled by three shaft seals arranged in series such that reactor coolant leakage to the containment is essentially zero. Seal injection flow is directed to each reactor coolant pump via a seal water injection filter. It enters each pump through a connection on the thermal barrier flange. Here the flow splits; the major portion flows down the shaft to cool the bearing and enters the RCS, and the remainder flows up the shaft through the seals. This seal flow provides a backpressure on the No. 1 seal and a controlled flow through the seal. After passing through the No. 1 seal, most of the flow leaves the pump via the No. 1 seal leakoff line. Minor flow passes through the No. 2 seal to its leakoff line. A backflush injection from a head tank flows into the No. 3 seal between its double dam seal area. At this point the flow divides, with half flushing through one side of the seal and out the No. 2 seal leakoff, while the remaining half flushes through the other side and out the No. 3 seal leakoff. This arrangement ensures essentially zero leakage of reactor coolant or trapped gases to the containment.

In the event of a loss of seal injection and ACCW flow to the thermal barrier heat exchanger, reactor coolant begins to travel along the RCP shaft and displace the cooler seal injection water. Once the temperature of the No. 1 seal reaches 250°F to 300°F, the SDS actuates by retraction of a thermal actuator, causing the SDS piston and polymer rings to constrict around the shaft. SDS actuation controls shaft seal leakage and limits the loss of reactor coolant through the RCP seal package.

5.4.1.3.11 Seal Discharge Piping

The No. 1 seal reduces the leakoff pressure to that of the volume control tank. Water from each pump No. 1 seal is piped to a common manifold, through the seal water return filter, and through the seal water heat exchanger where the temperature is reduced to that of the volume control tank. The No. 2 and 3 leakoff lines route No. 2 and 3 seal leakage to the reactor coolant drain tank and the containment sump, respectively.

5.4.1.4 <u>Tests and Inspections</u>

The reactor coolant pumps can be inspected in accordance with the ASME Code, Section XI, for inservice inspection of nuclear RCSs.

The pump casing is cast in one piece. Support feet are cast integrally with the casing to eliminate a weld region.

The design enables disassembly and removal of the pump internals for visual access to the pump casing.

The reactor coolant pump quality assurance program is given in table 5.4.1-2.

Tests and inspections performed under the following license renewal aging management programs are credited as applicable to the reactor coolant pumps and their subcomponents:

- Bolting Integrity Program (see subsection 19.2.2).
- Boric Acid Corrosion Control Program (see subsection 19.2.3).
- Inservice Inspection Program (see subsection 19.2.13).
- Water Chemistry Control Program (see subsection 19.2.28).

5.4.1.5 Pump Flywheel

The integrity of the reactor coolant pump flywheel is ensured on the basis of the following design and quality assurance procedures^a.

5.4.1.5.1 Design Basis

The calculated stresses at operating speed are based on stresses due to centrifugal forces. The stress resulting from the interference fit of the flywheel on the shaft is less than 2000 psi at zero speed, but this stress becomes zero at approximately 600 rpm because of radial expansion of the hub. The reactor coolant pumps run at approximately 1190 rpm and may operate briefly at overspeeds up to 109% (1295 rpm) during loss of offsite electrical power. For conservatism, however, 125% of operating speed was selected as the design speed for the reactor coolant pumps. The flywheels are given a manufacturer's test of 125% of the maximum synchronous speed of the motor.

5.4.1.5.2 Fabrication and Inspection

The flywheel consists of two thick plates bolted together. The flywheel material is produced by a process that minimizes flaws in the material and improves its fracture toughness properties, i.e., an electric furnace with vacuum degassing. Each plate is fabricated from SA-533, Grade B, Class 1 steel. Supplier certification reports are available for all plates and demonstrate the acceptability of the flywheel material on the basis of the requirements of NRC Regulatory Guide 1.14.

Flywheel blanks are flame cut from the SA-533, Grade B, Class 1 plates with at least 1/2 in. of stock left on the outer surface and bore surface for machining to final dimensions. The finished machined bores, keyways, and drilled holes are subjected to magnetic particle or liquid penetrant examinations in accordance with the requirements of Section III of the ASME Code. The finished flywheels, as well as the flywheel material (rolled plate), are subjected to 100% volumetric ultrasonic inspection using procedures and acceptance standards specified in Section III of the ASME Code.

^a Reactor coolant pump flywheel fatigue is evaluated as a time-limited aging analysis (TLAA) for license renewal (see paragraph 19.4.2.3).

The reactor coolant pump motors are designed such that, by removing the cover to provide access, the flywheel is available to allow an inservice inspection program which was originally in accordance with the recommendations of Regulatory Guide 1.14, referencing Section XI of the ASME Code for inspection scheduling purposes. Subsequently, the NRC authorized the use of an alternative flywheel inspection as addressed in Westinghouse WCAP-14535A, "Topical Report on Reactor Coolant Pump Flywheel Inspection Elimination." In this alternative inspection program, each flywheel is inspected at least once every 10 years by conducting either: (1) an inplace ultrasonic examination over the volume from the inner bore of the flywheel to the circle of one-half the outer radius, or (2) a surface examination (magnetic particle and/or liquid penetrant) of exposed surfaces of the disassembled flywheel.

5.4.1.5.3 Material Acceptance Criteria

The reactor coolant pump motor flywheel conforms to the following material acceptance criteria:

- A. The nil ductility transition temperature (NDTT) of the flywheel material is obtained by two drop weight tests which exhibit no-break performance at 20°F in accordance with ASTM E-208. The above drop weight tests demonstrate that the NDTT of the flywheel material is no higher than 10°F.
- B. A minimum of three Charpy V-notch (C_V) impact specimens from each plate shall be tested at ambient (70°F) temperature in accordance with the ASME SA-370 specification. The Charpy V-notch (C_V) energy in both the parallel and normal orientation with respect to the final rolling direction of the flywheel plate material is at least 50 ft-lb and 35-mil lateral expansion at 70°F, and, therefore, the flywheel material has a reference nil ductility temperature (RT_{NDT}) of 10°F. An evaluation of flywheel overspeed has been performed which concludes that flywheel integrity will be maintained.⁽¹⁾

Thus, it is concluded that flywheel plate materials are suitable for use and can meet Regulatory Guide 1.14 acceptance criteria on the bases of suppliers' certification data. The degree of compliance with Regulatory Guide 1.14 is further discussed in section 1.9.

5.4.1.6 References

- 1. "Reactor Coolant Pump Integrity in LOCA," WCAP-8163, September 1973.
- 2. "Safety-Related Research and Development for Westinghouse Pressurized Water Reactor, Program Summaries; Winter 1977 Summer 1978," <u>WCAP-8768</u>, Revision 2, October 1978.
- 3. Westinghouse Owners Group Letter OG-110 to Nuclear Regulatory Commission, December 11, 1983.
- 4. Westinghouse Owners Group Letter OG-117 to Nuclear Regulatory Commission, March 9, 1983.
- 5. "Resolution of TMI Action Item II.K.3.5, Automatic Trip of Reactor Coolant Pumps," Nuclear Regulatory Commission Generic Letters 83-10c and d, February 8, 1983.
- 6. "Use of Westinghouse Shield Passive Shutdown Seal for Flex Strategies," TR-FSE-14-1-P, Revision 1, March 2014.

7. "PRA Model for the Generation III Westinghouse Shutdown Seal, "PWROG-14001-P, Revision 0, June 2014.

5.4.2 STEAM GENERATORS

5.4.2.1 Design Bases

Steam generator design data are given in table 5.4.2-1. Code classifications of the steam generator components are given in section 3.2. Although the American Society of Mechanical Engineers (ASME) classification for the secondary side is specified to be Class 2, the current philosophy is to design all pressure-retaining parts of the steam generator, and thus both the primary and secondary pressure boundaries, to satisfy the criteria specified in Section III of the ASME Code for Class 1 components. The design stress limits, transient conditions, and combined loading conditions applicable to the steam generator are discussed in subsection 3.9.1. Estimates of radioactivity levels anticipated in the secondary side of the steam generators during normal operation and the bases for the estimates are given in chapter 11. The accident analysis of a steam generator tube rupture is discussed in chapter 15.

A design objective of the internal moisture separation equipment is that moisture carryover should not exceed 0.25% by weight under the following conditions:

- A. Steady-state operation up to 100% of full-load steamflow with water at the normal operating level.
- B. Loading or unloading at a rate of 5% of full-power steamflow per minute in the range from 15 to 100% of full-load steamflow.
- C. A step load change of 10% of full power in the range from 15- to 100% full-load steamflow.

The water chemistry on the reactor side, selected to provide the necessary boron content for reactivity control, should minimize corrosion of reactor coolant system (RCS) surfaces. The effectiveness of the water chemistry of the steam side in affecting corrosion control is discussed in chapter 10. Compatibility of steam generator tubing with both primary and secondary coolants is discussed further in paragraph 5.4.2.4.3.

The steam generator is designed to minimize unacceptable damage from mechanical or flow-induced vibration. Tube support adequacy is discussed in paragraph 5.4.2.3.3. The tubes and tube sheet are analyzed and confirmed to withstand the maximum accident loading conditions as they are defined in subsection 3.9.1. Further consideration is given in paragraph 5.4.2.3.4 to the effect of tube-wall thinning on accident condition stresses.

5.4.2.2 Design Description

The steam generator is a Model F, vertical-shell, and U-tube evaporator, with integral moisture separating equipment. Figure 5.4.2-1 shows the model, indicating several of its design features which are described in the following paragraphs.

On the primary side, the reactor coolant flows through the inverted U-tubes, entering and leaving through nozzles located in the hemispherical bottom head of the steam generator. The head is divided into inlet and outlet chambers by a vertical divider plate extending from the apex of the head to the tube sheet.

Steam is generated on the shell side, flows upward, and exits through the outlet nozzle at the top of the vessel. Feedwater enters the steam generator at an elevation above the top of the Utubes through a feedwater nozzle or the auxiliary feed nozzle. Stratification and striping in the main nozzle region is reduced by utilization of an auxiliary feedwater nozzle during startup, hot standby, and power escalation. During startup, the auxiliary feedwater system supplies the steam generator via the auxiliary nozzle. This mode continues up to about 4% power. At this level the switch is made to the main feedwater pumps which continue to feed into the auxiliary nozzle. Between 12- and 20% power, the main feed pumps feed into the main feed nozzle. The flowrate and temperature of the feedwater above 12% power should be high enough to substantially reduce stratification and striping loads on the main feed nozzle. During normal operation, the bypass line isolation valve is normally open, therefore, some flow will always be directed to the auxiliary nozzle. During hot standby the steam generator is supplied by the auxiliary feed system via the auxiliary nozzle. The water entering through the main feed nozzle is distributed circumferentially around the steam generator by means of a feedwater ring and then flows through an annulus between the tube wrapper and shell. The feedwater enters the ring via a welded thermal sleeve connection and leaves it through inverted J-tubes located at the flow holes which are at the top of the ring. The J-tubes are arranged to distribute the bulk of the colder feedwater to the hot leg side of the tube bundle. The feed ring is designed to minimize conditions which can result in water hammer occurrences in the feedwater piping. Water that enters through the auxiliary feed nozzle is not distributed through a feed ring. At the bottom of the wrapper, the water is directed toward the center of the tube bundle by a flow distribution baffle. This baffle arrangement serves to minimize the tendency in the relatively low-velocity fluid for sludge deposition. Flow-blocking devices discourage the water from flowing up the bypass lane as it enters the tube bundle where it is converted to a steam-water mixture. Subsequently, the steam-water mixture from the tube bundle rises into the steam drum section, where 16 individual centrifugal moisture separators remove most of the entrained water from the steam. The steam continues to the secondary separators for further moisture removal, increasing its quality to a designed minimum of 99.75%. The moisture separators divert the separated water, which is combined with entering feedwater to flow back down the annulus between the wrapper and shell for recirculation through the steam generator. The dry steam exits from the steam generator through the outlet nozzle, which is provided with a steamflow restrictor. (Refer to subsection 5.4.4.) The moisture carryover is expected to be well below the internal moisture separator design objective of 0.25 weight% for the cases with a primary T_{avg} at the high end of the proposed range of 570.7°F to 588.4°F.

With the T_{avg} at the lower end of the range, moisture carryover may exceed 0.25%. However, even for the low T_{avg} condition, moisture carryover will remain below 0.5%, which is the threshold beyond which erosion and corrosion of the piping and valves downstream of the steam generators are of a concern.

5.4.2.3 <u>Design Evaluation</u>

5.4.2.3.1 Forced Convection

The effective heat transfer coefficient is determined by the physical characteristics of the Model F steam generator and the fluid conditions in the primary and secondary systems for the nominal 100% design case. It includes a conservative allowance for fouling and uncertainty. A designed heat transfer area is provided to permit the achievability of the full-design heat-removal rate.

5.4.2.3.2 Natural Circulation Flow

In the event of loss of offsite power and consequential loss of forced circulation within the RCS, natural circulation functions to remove core decay heat and permit the plant to be stabilized in the hot standby operational mode. Under this condition, pressurizer pressure is maintained by one pressurizer backup heater group, which is powered from one of the emergency electrical buses. To ensure that one backup heater group is available, assuming a single failure, the VEGP is designed with the capability for manual loading of separate backup heater groups (i.e., group A and group B, respectively) on independent emergency, electrical buses (i.e., train A and train B, respectively) within 1 h, following loss of offsite power. One heater group (485 kW Unit 1 group A, 415 kW Unit 2 group A, 485 kW Unit 1 and Unit 2 group B) loaded within 1 h is sufficient to satisfy the minimum heat capacity requirement (150 kW) for natural circulation following loss of offsite power. This minimum heat capacity requirement conservatively covers the pressurizer heat losses at or below normal operating pressure, following loss of offsite power, and permits pressurizer pressure to be stabilized and maintained at any desired value.

The pressurizer heater design is such that following loss of offsite power and assuming a single failure, sufficient heater capacity is available to stabilize pressurizer pressure and preclude boiling in the RCS. If pressurizer heaters are not available to maintain pressurizer pressure, the RCS could be cooled via secondary side steam release. This operation would prevent saturation pressure from being reached in the active portion of the RCS. Depending on the circumstances, saturation conditions could occur in the upper head of the reactor vessel which would lead to the formation of a steam bubble. This would not impede natural circulation flow, however, since any vapor that entered the hot legs and subsequently the steam generators would be condensed by heat transfer to the secondary side of the steam generators. Vapor would be condensed in this manner as long as the steam generator tube bundle remains submerged.

5.4.2.3.3 Mechanical and Flow-Induced Vibration Under Normal Operating Conditions

In the design of the steam generators, the possibility of degradation of tubes due to either mechanical or flow-induced excitation is thoroughly evaluated. This evaluation includes detailed analysis of the tube support systems as well as an extensive research program with tube vibration model tests.

In evaluating degradation due to vibration, consideration is given to sources of excitation such as those generated by primary fluid flowing within the tubes, mechanically-induced vibration, and secondary fluid flow on the outside of the tubes. During normal operation, the effects of primary fluid flow within the tubes and mechanically-induced vibration are considered to be negligible and should cause little concern.

Thus, the primary source of tube vibrations is the hydrodynamic excitation by the secondary fluid on the outside of the tubes. In general, three vibration mechanisms have been identified:

- Vortex shedding.
- Fluidelastic excitation.
- Turbulence.

Vortex shedding does not provide detectable tube bundle vibration. There are several reasons why this happens.

- A. Flow turbulence in the downcomer and tube bundle inletregion inhibit the formation of Von Karman's vortex train.
- B. The spatial variations of crossflow velocities along the tube preclude vortex shedding at a single frequency.
- C. Both axial flow and crossflow velocity components exist on the tubes. The axial flow component disrupts the Von Karman vortices.

Fluid elastic excitation was observed during the testing. The amplitudes of the vibrations were two orders of magnitude smaller than those of the turbulent flow-induced vibrations. Therefore, fluid elastic excitation is excluded from consideration as a factor in steam generator tube bundle vibrations.

Flow-induced vibrations due to flow turbulence cause stresses in the tubes that are two orders of magnitude below the endurance limit (30,000 psi) of the tube material. Therefore, the contribution to fatigue is negligible; and fatigue degradation from flow-induced vibration is not anticipated during normal operation.

Summarizing the results of analyses and tests of the steam generator for vibration, it can be stated that a check of all modes of tube vibration mechanisms has been completed. The conclusions that can be drawn are that the primary source of tube vibration is fluid turbulence and that the magnitude of the vibration is so small that, when combined with its total random nature, its contribution to tube fatigue is negligible. Therefore, fatigue degradation due to flow-induced vibration is not anticipated.

The impact of operation at a power level of 3653 MWt has been evaluated and it is concluded that significant levels of vibration will not occur from the fluid-elastic, vortex shedding, or turbulent mechanisms. The projected level of tube wear as a result of vibration will remain small and will not result in unacceptable tube wear.

5.4.2.3.4 Allowable Tube-Wall Thinning Under All Plant Conditions^a

An analysis has been performed to define the structural limits for an assumed uniform thinning mode of degradation in both the axial and circumferential directions at a power level of 3653 MWt. The assumption of uniform thinning is generally regarded to result in a conservative structural limit for all flaw types occurring in the field. The allowable tube repair limit, in accordance with Regulatory Guide 1.121, is obtained by incorporating into the structural limit a growth allowance for continued operation until the next scheduled inspection, as well as an allowance for eddy current measurement uncertainty.

Calculations have been performed to establish the tube straight leg (free span) region of the tube for degradation over an unlimited axial length, and for degradation over limited axial extent at the tube support plate (TSP), the flow distribution baffle (FDB), and antivibration bar intersections for the 3653 MWt power level conditions.

The minimum structural limit is calculated to be 60.0% allowable tube wall loss for the high / low T_{avg} straight length location. The straight leg structural limit is also applicable to the tube / AVB tangent points. These tube / AVB tangent points correspond to row 7 for the inner sets of AVBs, row 20 for the middle sets of AVBs, and row 31 for the outer sets of AVBs. The enveloping structural limits at the FDB, TSP, and anti-vibration bars at high / low T_{avg} (other than the tube / AVB tangent points) are 66.5%, 62.0%, and 72.0%, respectively.

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^a Loss of material from the steam generator tube walls was evaluated as a TLAA for license renewal in accordance with 10 CFR Part 54. The results of this evaluation are provided in paragraph 19.4.6.3.

Structural integrity performance criteria (SIPC) requirements were evaluated to address circumferential cracks at the TSP and U-bend regions.

Results are provided in% degraded area (reference 3).

5.4.2.4 <u>Steam Generator Materials</u>

5.4.2.4.1 Selection and Fabrication of Materials

All pressure boundary materials used in the steam generator are selected and fabricated in accordance with the requirements of Section III of the ASME Code. A general discussion of materials specifications is given in subsection 5.2.3, with types of materials listed in tables 5.2.3-1 and 5.2.3-2. Fabrication of reactor coolant pressure boundary (RCPB) materials is also discussed in subsection 5.2.3, particularly in paragraphs 5.2.3.3 and 5.2.3.4

Testing has justified the selection of corrosion-resistant Inconel-600, a nickel-chromium-iron alloy (ASME SB-163), for the steam generator tubes. The channel head divider plate is Inconel (ASME SB-168). The interior surfaces of the reactor coolant channel head, nozzles, and manways are clad with austenitic stainless steel. The primary side of the tube sheet is weld clad with Inconel (ASME SFA-5.14). The tubes are then seal welded to the tube-sheet cladding. These fusion welds, performed in compliance with Sections III and IX of the ASME Code, are dye-penetrant inspected and leakproof tested before each tube is hydraulically expanded the full depth of the tube-sheet bore.

Code cases used in material selection are discussed in subsection 5.2.1. The extent of conformance with Regulatory Guides 1.84, Design and Fabrication Code Case Acceptability ASME Section III, Division 1, and 1.85, Materials Code Case Acceptability ASME Section III, Division 1, is discussed in section 1.9.

During manufacture, cleaning is performed on the primary and secondary sides for the steam generator in accordance with written procedures which follow the guidance of Regulatory Guide 1.37, Quality Assurance Requirements for Cleaning of Fluid Systems and Associated Components of Water-Cooled Nuclear Power Plants, and American National Standards Institute (ANSI) Standard N45.2.1-1973, Cleaning of Fluid Systems and Associated Components for Nuclear Power Plants. Onsite cleaning and cleanliness control also follow the guidance of Regulatory Guide 1.37 as discussed in section 1.9. Cleaning process specifications are discussed in paragraph 5.2.3.4.

The fracture toughness of the materials is discussed in paragraph 5.2.3.3. Adequate fracture toughness of ferritic materials in the RCPB is provided by compliance with 10 CFR 50, Appendix G, Fracture Toughness Requirements, and Paragraph NB-2300 of Section III of the ASME Code, and by meeting the requirements of General Design Criteria 1, 14, 15, and 31.

5.4.2.4.2 Steam Generator Design Effects on Materials

Several features have been introduced into the Model F steam generator to minimize the deposition of contaminants from the secondary-side flow. Such deposits could otherwise produce a local environment in which adverse conditions could develop and result in material corrosion. The support plates are made of corrosion resistant stainless steel 405 alloy and incorporate a four-lobe-shaped tube hole design that provides greater flow area adjacent to the tube outer surface and eliminates the need for interstitial flow holes. The resulting increase in

flow provides higher sweeping velocities at the tube/tube support plate intersections. These increased sweeping velocities reduce the potential for sludge deposition on the tube support plates. Historically, sludge removal from the tube support plates has not been performed. Figure 5.4.2-2 is an illustration of the guatrefoil broached holes. This modification in the support plate design is a major factor contributing to the increased circulation ratio. The increased circulation results in increased flow in the interior of the bundle, as well as increased horizontal velocity across the tube sheet, which reduces the tendency for sludge deposition. The effect of the increased circulation on the vibrational stability of the tube bundle has been analyzed with consideration given to flow-induced excitation frequencies. The unsupported span length of tubing in the U-bend region and the corresponding optimum number of antivibration bars has been determined. The antivibration bars are fabricated from square Inconel barstock which is then chromium treated to improve frictional characteristics. Because of the increased circulation ratio, the moisture separating equipment has been modified to maintain an adequate margin with respect to the moisture carryover. To provide added strength as well as resistance to vibration, the quatrefoil tube support plate thickness has been increased. In addition, either 8 or 12 peripheral supports provide stability to the plates so that tube fretting or wear due to flowinduced plate vibrations at the tube support contact regions is minimized. For the top tube support plate, there is a continuous ring to distribute the lateral loads.

Assurance against significant flow-induced tube vibration is provided by a combination of analysis and testing.

Combining both vortex shedding and turbulence effects in a conservative manner, the maximum predicted local tube wear depth over a 40-year operating design objective^a remains less than 0.006 inch with the operation at a power level of 3653 MWt. This value is considerably below the plugging limit for Model F steam generators.

5.4.2.4.3 Compatibility of Steam Generator Tubing With Primary and Secondary Coolants^b

As mentioned in paragraph 5.4.2.4.1, corrosion tests which subjected the steam generator tubing material, Inconel-600 (ASME SB-163), to simulated steam generator water chemistry have indicated that the loss due to general corrosion over the 40-year operating design objective^a is insignificant compared to the tube-wall thickness. Testing to investigate the susceptibility of heat exchanger construction materials to stress corrosion in caustic and chloride aqueous solutions has indicated that Inconel-600 has resisted general corrosion in severe operating water conditions. Many reactor years of successful operation have shown the same low general corrosion rates as indicated by the laboratory tests.

Recent operating experience, however, has revealed areas on secondary surfaces where localized corrosion rates were significantly greater than the low general corrosion rates. Both intergranular stress corrosion and tube-wall thinning were experienced in localized areas, although not simultaneously at the same location or under the same environmental conditions (water chemistry, sludge composition).

^a The operating licenses for both VEGP units have been renewed and the original licensed operating terms have been extended by 20 years. In accordance with 10 CFR Part 54, appropriate aging management programs and activities have been initiated to manage the detrimental effects of aging to maintain functionality during the period of extended operation (see chapter 19).

^b The Water Chemistry Control Program is credited as a license renewal aging management program (see subsection 19.2.28).

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The adoption of the all volatile treatment (AVT) control program minimizes the possibility for recurrence of the tube-wall thinning phenomenon. Successful AVT operation requires maintenance of low concentrations of impurities in the steam generator water, thus reducing the potential for formation of highly concentrated solutions in low-flow zones, which is the precursor of corrosion. By restriction of the total alkalinity in the steam generator and prohibition of extended operation with free alkalinity, the AVT program should minimize the possibility for recurrence of intergranular corrosion in localized areas due to excessive levels of free caustic.

Laboratory testing has shown that the Inconel-600 tubing is compatible with the AVT environment. Isothermal corrosion testing in high-purity water has shown that commercially produced Inconel-600 exhibiting normal microstructures tested at normal engineering stress levels does not suffer intergranular stress corrosion cracking in extended exposure to high-temperature water. These tests also showed that no general type corrosion occurred. A series of autoclave tests in reference secondary water with planned excursions has produced no corrosion attack after 1938 days of testing on any as-produced Inconel-600 tube samples.

Model boiler tests have been used to evaluate the AVT chemistry guidelines adopted in 1974. The guidelines appear to be adequate to preserve tube integrity with one significant alteration: operation with contaminant ingress must be limited.

Additional extensive operating data are presently being accumulated with the conversion to AVT chemistry. A comprehensive program of steam generator inspections, including the recommendations of Regulatory Guide 1.83, Inservice Inspection of Pressurized Water Reactor Steam Generator Tubes, with the exceptions as stated in section 1.9, should provide for detection of any degradation that might occur in the steam generator tubing. Action levels for secondary side water chemistry during power operation are given in the EPRI PWR Secondary Water Chemistry Guidelines.

Increased margin against stress corrosion cracking has been obtained by the use of thermally treated Inconel-600 tubing. Thermal treatment of Inconel tubes has been shown to be particularly effective in resisting caustic corrosion. Tubing used in the Model F is thermally treated in accordance with a laboratory derived treatment process.

The tube support plates used in the Model F are ferritic stainless steel, which has been shown in laboratory tests to be resistant to corrosion in the AVT environment. If corrosion of ferritic stainless steel were to occur due to concentration of contaminants, the volume of the corrosion products is essentially equivalent to the volume of the parent material consumed. This would be expected to preclude denting. The support plates are also designed with quatrefoil tube holes rather than cylindrical holes. The quatrefoil tube-hole design promotes high-velocity flow along the tube and is expected to minimize the accumulation of impurities at the support plate location.

Additional measures are incorporated in the Model F design to prevent areas of dryout in the steam generator and accumulations of sludge in low-velocity areas. Modifications to the wrapper have increased water velocities across the tube sheet. A flow distribution baffle is provided which forces the low-flow area to the center of the bundle. Increased capacity blowdown pipes have been added to enable continuous blowdown of the steam generators at a high volume. The intakes of these blowdown pipes are located below the center cutout section of the flow distribution baffle in the low-velocity region where sludge may be expected to accumulate.

The impact of operating at a power level of 3653 MWt on steam generator water chemistry has been considered. The occurrence of stress corrosion cracking (SCC) and other forms of degradation that might occur at current and enhanced rates will be found using the

nondestructive examination (NDE) techniques specified in the degradation assessment that must be completed for subsequent plant outages.

5.4.2.4.4 Cleanup of Secondary-Side Materials

Several methods are employed to clean operating steam generators of corrosion-causing secondary-side deposits. Sludge lancing, a procedure in which a hydraulic jet inserted through an access opening (handhole) loosens deposits, can be performed when the need is indicated by the results of steam generator tube inspection. Six 6-in. access ports are provided for sludge lancing and inspection. Three of these are located above the tube sheet and three above the flow distribution baffle. Continuous blowdown is performed to monitor water chemistry. The location of the blowdown piping suction, adjacent to the tube sheet and in a region of relatively low-flow velocity, facilitates the removal of particulate impurities to reduce the accumulation on the tube sheet.

5.4.2.5 Steam Generator Inservice Inspection^a

The steam generator is designed to permit inspection of Class 1 and 2 parts, including individual tubes. The design includes a number of openings to provide access to both the primary and secondary sides of the steam generator. The preservice inspection of the Unit 1 steam generators was completed to the requirements of the 1980 ASME Code through Winter 1980 Addenda. Inservice inspection will comply with the requirements of 10 CFR 50.55a. The openings include four manways, two for access to both chambers of the reactor coolant channel head inlet and outlet sides and two in the steam drum for inspection and maintenance of the moisture separators, and eight 6-in. handholes, three located just above the tube-sheet secondary surface, three located just above the flow distribution baffle, and two located on the tubelane diameter between the upper tube support plate and the row 1 tubes. Additional access to the tube U-bend is provided through each of the three deck plates. For proper functioning of the steam generator, some of the deck-plate openings are covered with welded, but removable, hatch plates. Inspection/access to the primary side is provided by two 16-in. manways located in the channel head.

Regulatory Guide 1.83 provides recommendations concerning the inspection of tubes, which cover inspection equipment, baseline inspections, tube selection, sampling and frequency of inspection, methods of recording, and required actions based on findings. The steam generators are designed to permit access to tubes for inspection and/or repair or plugging, if necessary, per the guidelines described in Regulatory Guide 1.83. Regulatory Guide 1.121, Basis for Plugging Degraded Pressurized Water Reactor (PWR) Steam Generator Tubes, provides recommendations concerning tube plugging. The minimum requirements for inservice inspection of steam generators, including tube plugging criteria, are established as part of the Technical Specifications.

5.4.2.6 Quality Assurance

The steam generator quality assurance program is given in table 5.4.2-2.

^a The Inservice Inspection Program, Steam Generator Tubing Integrity Program, and Steam Generator Program for Upper Internals are credited as license renewal aging management programs (see subsections 19.2.13, 19.2.26, and 19.2.27).

Radiographic inspection and acceptance standard are in accordance with the requirements of Section III of the ASME Code.

Liquid penetrant inspection is performed on weld-deposited tube-sheet cladding, channel-head cladding, divider-plate-to-tube-sheet and to channel-head weldments, tube-to-tube-sheet weldments, and weld-deposit cladding. Liquid penetrant inspection and acceptance standards are in accordance with the requirements of Section III of the ASME Code.

Some of the tube-to-tube sheet welds of Unit 1 steam generator number 4 were deformed by impacts from a loose part. Based on the completion of a testing and analysis program, it is concluded that tube-to-tubesheet joint integrity remains consistent with the original plant design basis. The minimum requirements for the inservice inspection of steam generator tubes, including tube plugging criteria, are established as part of the Technical Specifications and are not changed as a result of the mechanical deformation of the tube-to-tubesheet welds.

Magnetic particle inspection is performed on the tube-sheet forging, channel-head casting, nozzle forgings, and the following weldments:

- A. Nozzle to shell.
- B. Support brackets.
- C. Instrument connection (secondary).
- D. Temporary attachments after removal.
- E. All accessible pressure-retaining welds after hydrostatic test.

Magnetic particle inspection and acceptance standards are in accordance with the requirements of Section III of the ASME Code.

Ultrasonic tests are performed on the tube-sheet forgings, tube-sheet cladding, secondary-shell and heat plates, and nozzle forgings.

The heat transfer tubing is subjected to eddy current testing and ultrasonic examination.

Hydrostatic tests are performed in accordance with Section III of the ASME Code.

5.4.2.7 References

1. <u>WCAP-16543-P</u>, Rev. 0, "Regulatory Guide 1.121 Analysis and Structural Integrity Performance Criterion Application for the Vogtle Units 1 & 2 Model F Steam Generators," August 2006.

5.4.3 REACTOR COOLANT PIPING

5.4.3.1 <u>Design Bases</u>

The reactor coolant system (RCS) piping is designed and fabricated to accommodate the system pressures and temperatures attained under all expected modes of plant operation or anticipated system interactions. Stresses are maintained within the limits of Section III of the American Society of Mechanical Engineers (ASME) Code. Code and material requirements are provided in section 5.2.

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Materials of construction are specified to minimize corrosion/ erosion and ensure compatibility with the operating environment.

The piping in the RCS is Safety Class 1 and is designed and fabricated in accordance with ASME Code, Section III, Class 1 requirements.

Stainless steel pipe conforms to American National Standards Institute (ANSI) B36.19 for sizes 1/2 in. through 12 in. and wall thickness schedules 40S through 80S. Stainless steel pipe outside of the scope of ANSI B36.19 conforms to ANSI B36.10.

The minimum wall thickness of the loop piping and fittings are no less than those calculated using the ASME Code, Section III, Class 1 formula of Paragraph NB-3641.1(3) with an allowable stress value of 17,550 psi. The pipe wall thickness for the pressurizer surge line is schedule 160. The minimum pipe bend radius is 5 nominal pipe diameters, and ovality does not exceed 6%.

All butt welds, branch connection nozzle welds for 3 in. nominal pipe sizes and greater, and boss welds are of a full penetration design. Socket-welded connections could be used for branch connection nozzle welds for 2 in. nominal pipe sizes and smaller and for some thermowell connections.

Processing and minimization of sensitization are discussed in subsection 5.2.3.

Flanges conform to ANSI B16.5.

Socket weld fittings and socket joints conform to ANSI B16.11.

Inservice inspection is discussed in subsection 5.2.4.

5.4.3.2 <u>Design Description</u>

The RCS piping includes those sections of piping interconnecting the reactor vessel, steam generator, and reactor cooling pump (RCP). It also includes the following:

- A. Charging line and alternate charging line from the designated check valve up to the branch connections on the reactor coolant loop (RCL).
- B. Letdown line and excess letdown line from the branch connections on the RCL to the system isolation valve.
- C. Pressurizer spray lines from the reactor coolant cold legs to the spray nozzle on the pressurizer vessel.
- D. Residual heat removal (RHR) lines to or from the RCLs up to the designated check valve or isolation valve.
- E. Safety injection lines from the designated check valve to the RCLs.
- F. Accumulator lines from the designated check valve to the RCLs.
- G. Deleted.
- H. Loop, fill, drain^(b), sample, and instrument^(a) lines from the designated isolation valve to the RCLs.
- I. Pressurizer surge line from one RCL hot leg to the pressurizer vessel surge nozzle.

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- J. Resistance temperature detector scoop element, pressurizer spray scoop, sample connection^(a) with scoop, reactor coolant temperature element installation boss, the temperature element well itself, and the resistance temperature detector (RTD) fast response thermowells.
- K. All branch connection nozzles attached to RCLs.
- L. Pressure safety and relief lines from nozzles on top of the pressurizer vessel up to and through the pressurizer power-operated relief valves and pressurizer safety valves.
- M. Auxiliary spray line from the isolation valve to the main pressurizer spray line.
- N. Sample lines^(a) from the pressurizer to the isolation valve.
- O. Vent line from the reactor vessel head to the system isolation valves.
- P. Reactor vessel level instrumentation lines from the isolation valves to the reactor vessel and to two RCL hot legs.
- Q. RCP seal water injection lines to or from the RCP.
- R. Boron injection lines from the check valve to the RCL.

Principal design data for the reactor coolant piping are given in table 5.4.3-1.

Details of the materials of construction and codes used in the fabrication of reactor coolant piping and fittings are discussed in section 5.2.

The reactor coolant piping and fittings that make up the loops are austenitic stainless steel. Pipe and fittings are cast, seamless without longitudinal or electroslag welds, and comply with the requirements of the ASME Code, Section II (parts A and C), Section III, and Section IX. All smaller piping that is part of the RCS, such as the pressurizer surge line, spray and relief line, loop drains^(b) and connecting lines to other systems, are also austenitic stainless steel. The nitrogen supply line for the pressurizer relief tank is carbon steel. All joints and connections are welded, except for the pressurizer code safety valves, where flanged joints are used.

All piping connections from auxiliary systems are above the horizontal centerline of the reactor coolant piping, with the exception of:

- A. RHR pump suction lines, which are 45° down from the horizontal centerline. This enables the water level in the RCS to be lowered in the reactor coolant pipe while continuing to operate the RHRS, should this be required for maintenance.
- B. Loop drain lines^(b) and the connection for temporary level measurement of water in the RCS during refueling and maintenance operation.
- C. The differential pressure taps for flow measurement, which are downstream of the steam generators of the first 90° elbow.
- D. The pressurizer surge line, which is attached at the horizontal centerline.
- E. Two of the three thermowells in each resistance temperature detector hot leg connection.

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^(a) Lines with a 3/8-in. or less flow restricting orifice qualify as Safety Class 2. In the event of a break in one of these Safety Class 2 lines, the normal makeup system is capable of providing makeup flow while maintaining pressurizer water level.

⁽b) (Unit 1 only) Flow path from Loop 3 RCL intermediate leg to liquid waste processing system drain header into the reactor coolant drain tank has been removed.

Penetrations into the coolant flow path are limited to the following:

- A. The spray line inlet connections extend into the cold leg piping in the form of a scoop so that the velocity head of the RCL flow adds to the spray driving force.
- B. The RCS sample taps protrude into the main stream to obtain a representative sample of the reactor coolant.
- C. The hot and cold leg thermowells for the resistance temperature detectors extend into the reactor coolant to measure the RCS temperatures. For the hot legs, the RTDs are located inside of the RTD scoops.
- D. The wide-range temperature detectors are located in resistance temperature detector wells that extend into both the hot and cold legs of the reactor coolant piping.

Separate RTDs mounted in thermowells in each RCL hot and cold leg are provided so that individual temperature signals may be developed for use in the reactor control and protection systems. The RTDs are contained in thermowells which extend into the hot leg flow to become exposed to a representative temperature sample of the reactor coolant. Each hot leg is instrumented with three thermowells at locations 120 degrees apart around the reactor coolant piping with one located at the top of the pipe. The temperature measured by the three RTDs is then averaged using electronic weighting to provide the temperature input to the reactor control and protection systems.

An RTD mounted in a thermowell for the cold leg temperature measurement is located downstream of each reactor coolant pump discharge. This connection is located close to the same weld connection at the pump discharge and is in the same relative position in each loop.

Signals from the temperature detectors are used to compute the reactor coolant ΔT (temperature of the hot leg, T_{hot} , minus the temperature of the cold leg, T_{cold}), and an average reactor coolant temperature, T_{avg} . The ΔT and T_{avg} for each loop is indicated on the main control board.

5.4.3.3 Design Evaluation

Piping load and stress evaluation for normal operating loads, seismic loads, blowdown loads, and combined normal, blowdown, and seismic loads is discussed in section 3.9.N.

5.4.3.3.1 Material Corrosion/Erosion Evaluation

The water chemistry is selected to minimize corrosion. A periodic analysis of the coolant chemical composition is performed to verify that the reactor coolant quality meets the specifications. (See subsection 5.2.3.)

Periodic analysis of the coolant chemical composition is performed to monitor the adherence of the system to desired reactor coolant water quality listed in table 5.2.3-3. Maintenance of the water quality to minimize corrosion is accomplished using the CVCS and sampling system which are described in chapter 9.

The design and installation are in compliance with the ASME Code, Section III. Pursuant to this, all pressure-containing welds out to the second valve that delineates the RCS boundary are accessible for inservice examination as required of ASME Code, Section XI, and are fitted with removable insulation.

5.4.3.3.2 Sensitized Stainless Steel

Sensitized stainless steel is discussed in subsection 5.2.3.

5.4.3.3.3 Contaminant Control

Contamination of stainless steel and Inconel by copper, low melting temperature alloys, mercury, and lead is prohibited. Colloidal graphite is the only permissible thread lubricant.

Prior to application of thermal insulation, the austenitic stainless steel surfaces are cleaned and analyzed to a halogen limit of 0.0015 mg chloride/dm² and 0.0015 mg fluoride/dm².

5.4.3.3.4 Detection Of Thermal Stratification^a

To assure that unisolable sections of piping connected to the reactor coolant system (RCS) and pressurizer surge line will not be subjected to combined cyclic and thermal stresses that could cause fatigue failure, a program has been implemented that will detect adverse temperature distributions as described below.

5.4.3.3.4.1 <u>Unisolable Section of Piping Connected To The RCS</u>. Unisolable sections of piping for the safety injection, normal and alternate charging, and auxiliary spray lines interconnected with the RCS are instrumented to detect adverse thermal stratification and cycling due to potential isolation valve leakage into the RCS boundary. Fluid leakage is detected by temperature measurements utilizing resistance temperature detectors (RTDs) strapped on the pipe. Temperature data is periodically recorded and evaluated for thermal stratification and cycling to determine its impact on piping structural integrity. Additionally (on Unit 2 only), two 12-in. residual heat removal (RHR) suction lines attached to the reactor coolant loop (RCL) hot leg are instrumented with RTDs.

5.4.3.3.4.2 <u>Pressurizer Surge Line</u>. The Unit 2 pressurizer surge line was instrumented to detect temperature distribution and thermal movement. The instruments consisted of RTDs and displacement transducers. The temperature distribution and thermal movements were monitored and recorded through the first refueling cycle. Analysis utilizing the leak-before-break methodology and the monitoring data taken from the Unit 2 surge line, combined with Unit 1 surge line support design changes and operating procedure changes, are an acceptable program to address the effects of thermal stratification on the pressurizer surge line. Therefore, no further monitoring is required.

5.4.3.4 Tests and Inspections

The RCS piping quality assurance program is given in table 5.4.3-2.

Volumetric examination is performed throughout 100% of the wall volume of each pipe and fitting in accordance with the applicable requirements of Section III of the ASME Code for all

^a The Fatigue Monitoring Program is credited as a license renewal aging management program (see subsection 19.3.2).

pipe 27 1/2 in. and larger. All unacceptable defects are eliminated in accordance with the requirements of the same section of the code.

A liquid penetrant examination is performed on all accessible surfaces of each finished fitting, in accordance with the criteria of the ASME Code, Section III. Acceptance standards are in accordance with the applicable requirements of the ASME Code, Section III.

The pressurizer surge line conforms to SA-376, Grade 304, 304N, or 316 with supplementary requirements S2 (transverse tension tests) and S6 (ultrasonic test). The S2 requirement applies to each length of pipe. The S6 requirement applies to 100% of the piping wall volume.

The end of pipe sections, branch ends, and fittings are machined back to provide a smooth weld transition adjacent to the weld path.

Tests and inspections performed under the following license renewal aging management programs are credited as applicable to various portions of RCS piping:

- Boric Acid Corrosion Control Program (see subsection 19.2.3).
- CASS RCS Fitting Evaluation Program (see subsection 19.2.5).
- External Surfaces Monitoring Program (see subsection 19.2.8).
- Inservice Inspection Program (see subsection 19.2.13).
- Oil Analysis Program (see subsection 19.2.16).
- One-Time Inspection Program (see subsection 19.2.17).
- One-Time Inspection Program for ASME Class 1 Small Bore Piping (see subsection 19.2.18).
- Water Chemistry Control Program (see subsection 19.2.28).
- Fatigue Monitoring Program (see subsection 19.3.2).

5.4.4 MAIN STEAM LINE FLOW RESTRICTIONS

5.4.4.1 Design Bases

The outlet nozzle of the steam generator is provided with a flow restrictor designed to limit steamflow in the unlikely event of a break in the main steam line. A large increase in steamflow will create a backpressure which limits further increase in flow. The flow restrictor performs the following functions:

- Rapid rise in containment pressure is limited.
- The rate of heat removal from the reactor is such as to keep the cooldown rate within acceptable limits.
- Thrust forces on the main steam line piping are reduced.

• Stresses on internal steam generator components, particularly the tube sheet and tubes, are limited.

The restrictor is configured to minimize the unrecovered pressure loss across the restrictor during normal operation.

5.4.4.2 Design Description

The flow restrictor consists of seven Inconel ASME SB-163 venturi inserts which are installed in holes in an integral steam outlet nozzle forging. The inserts are arranged with one venturi at the centerline of the outlet nozzle and the other six equally spaced around it. After insertion into the nozzle forging holes, the Inconel venturi inserts are welded to the Inconel cladding on the inner surface of the forging.

5.4.4.3 <u>Design Evaluation</u>

The flow restrictor design has been analyzed to determine its structural adequacy. The equivalent throat diameter of the steam generator outlet is 16 in. and the resultant pressure drop through the restrictor at 100% steamflow is approximately 2.78 psig. This is based on a design flowrate of 3.78×10^6 lb/h. Materials of construction and manufacturing of the flow restrictor are in accordance with Code Class 1 Section III of the ASME Code. The method for seismic analysis is dynamic.

5.4.4.4 <u>Inspections</u>

Since the restrictor is not part of the steam system boundary, no inspections beyond those performed during fabrication are anticipated.

5.4.5 MATERIALS AND INSPECTIONS

This subsection is not applicable to VEGP.

5.4.6 REACTOR VESSEL DESIGN DATA

This subsection is not applicable to VEGP.

5.4.7 RESIDUAL HEAT REMOVAL SYSTEM

The residual heat removal system (RHRS) transfers heat from the reactor coolant system (RCS) to the nuclear service cooling water system via the component cooling water (CCW) system to reduce the temperature of the reactor coolant to the cold shutdown temperature at a controlled rate during the second part of normal plant cooldown and maintains this temperature until the plant is started up again.

Refer to subsection 9.2.2 for a description of the CCW system. Parts of the RHRS also serve as parts of the emergency core cooling system (ECCS) for accident mitigation (section 6.3).

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In addition, the RHRS is used to transfer refueling water between the refueling cavity and the refueling water storage tank at the beginning and end of the refueling operations.

Nuclear plants employing the same RHRS design as the VEGP are given in section 1.3.

5.4.7.1 <u>Design Bases</u>

The RHRS design parameters are listed in table 5.4.7-1.

The following establishes a design bases supporting a 50°F/h cooling rate. The RHR system and support systems have been reviewed and evaluated to document the acceptability of an operational cooling rate of up to 100°F/h not to exceed 100°F in any 1 h period.

The RHRS is placed in operation approximately 2 to 4 hours after reactor shutdown, when the temperature and pressure of the RCS are approximately 350°F and 365 psig, respectively. Assuming that two heat exchangers and two pumps are in service and that each heat exchanger is supplied with 5000 gal/min CCW initially at 105°F, the RHRS is designed to reduce the temperature of the reactor coolant to 140°F within 20 h following reactor shutdown. Under these conditions, the time required to reduce the reactor coolant temperature from 350°F to 200°F is approximately 3 h. The heat load handled by the RHRS during the cooldown transient includes residual and decay heat from the core and reactor coolant pump heat. The design heat load is based on the decay heat fraction that exists at 20 h following reactor shutdown from an extended run at full power.

Assuming that only one heat exchanger and pump are in service and that the heat exchanger is supplied with CCW at 5000 gal/min and initially at 105°F, the RHRS is capable of reducing the temperature of the reactor coolant from 350°F to 200°F within approximately 30 h. The time required under these conditions to reduce reactor coolant temperature from 350°F to 212°F is approximately 20 h. The RHRS is designed to be isolated from the RCS whenever the RCS pressure exceeds the RHRS design pressure. The RHRS is isolated from the RCS on the suction side by two motor-operated valves in series on each suction line. Each motor-operated valve is interlocked to prevent its opening if RCS pressure is greater than approximately 365 psig. The valves have a control room alarm which alerts the operators if one or both of the valves is not fully closed and the RCS pressure exceeds 420 psig. The RHRS is isolated from the RCS on the discharge side by two check valves in each return line. Also provided on the discharge side is a normally open motor-operated valve downstream of each RHRS heat exchanger. (These check valves and motor-operated valves are not considered part of the RHRS; they are shown as part of the ECCS. See drawing 1X4DB121.)

Each inlet line to the RHRS is equipped with a pressure relief valve designed to prevent RHRS overpressurization assuming the most severe overpressure transients. These relief valves protect the system from inadvertent overpressurization during plant startup, shutdown, and cold shutdown decay heat-removal operations.

Each discharge line from the RHRS to the RCS is equipped with a pressure relief valve designed to relieve the maximum possible backleakage through the valves isolating the RHRS from the RCS. These valves are considered part of the ECCS, as depicted in drawing 1X4DB120. Relief capacity of these valves is given in table 6.3.2-2.

The RHRS is designed for a single nuclear power unit and is not shared between units.

The RHRS is designed to be fully operable from the control room for normal operation except as described in paragraph 5.4.7.2.7. By nature of its redundant design, the RHRS is designed to accept all major component single failures, with the only effect being an extension in the

required cooldown time. There are no motor-operated valves in the RHRS that are subject to flooding following a secondary side break or a LOCA. Although considered to be of low probability, spurious operation of a single motor-operated valve can be accepted without loss of function as a result of the redundant two-train design.

Missile protection, protection against dynamic effects associated with the postulated rupture of piping, and seismic design are discussed in section 3.5 and subsections 3.6.2 and 3.7.N.2, respectively.

5.4.7.2 System Design

5.4.7.2.1 Schematic Piping and Instrumentation Diagrams

The RHRS, as shown in drawings 1X4DB122 and 2X4DB122 and figure 5.4.7-1, consists of two residual heat exchangers, two RHR pumps, and the associated piping, valves, and instrumentation necessary for operational control. Notes to figure 5.4.7-1 identify modes of operation, valve alignments, and process conditions at various points in the system. The inlet lines to the RHRS are connected to the hot legs of two reactor coolant loops (RCLs), while the return lines are connected to the cold legs of each of the RCLs. These return lines are also the ECCS low-head injection lines (drawings 1X4DB119, 2X4DB119, 1X4DB120, and 1X4DB121). The RHRS suction lines are isolated from the RCS by two motor-operated valves in series located inside the containment. Each discharge line is isolated from the RCS by two check valves located inside the containment and by a normally open motor-operated valve, with power lockout capability, located outside the containment. (The check valves and the motor-operated valve on each discharge line are not part of the RHRS; these valves are shown as part of the ECCS (drawings 1X4DB119, 2X4DB119, 1X4DB120, and 1X4DB121).

During RHRS operation, reactor coolant flows from the RCS to the RHR pumps, through the tube side of the residual heat exchangers, and back to the RCS. The heat is transferred to the CCW circulating through the shell side of the residual heat exchangers.

Coincident with operation of the RHRS, a portion of the reactor coolant flow may be diverted from downstream of the residual heat exchangers to the chemical and volume control system (CVCS) low-pressure isolation letdown line for cleanup and/or pressure control. By regulating the diverted flowrate and the charging flow, the RCS pressure may be controlled. Pressure regulation is necessary to maintain the pressure range dictated by the fracture prevention criteria requirements of the reactor vessel and by the No. 1 seal differential pressure and net positive suction head requirements of the reactor coolant pumps.

The RCS cooldown rate is manually controlled by regulating the reactor coolant flow through the tube side of the residual heat exchangers. The flow control valve in the bypass line around each residual heat exchanger automatically maintains a constant return flow to the RCS. Instrumentation is provided to monitor system pressure, temperature, and total flow.

The RHRS is also used for filling the refueling cavity before refueling. After refueling operations, the RHRS is used to pump the water back to the refueling water storage tank until the water level is brought down to the flange of the reactor vessel. The remainder of the water is removed via a drain connection at the bottom of the refueling canal.

When the RHRS is in operation, the water chemistry is the same as that of the reactor coolant. Provision is made for the process sampling system to extract samples from the flow of reactor coolant downstream of the residual heat exchangers. A local sampling point is also provided on each RHR train between the pump and heat exchanger.

The RHRS also functions, in conjunction with the high-head portion of the ECCS, to provide injection of borated water from the refueling water storage tank into the RCS cold legs during the injection phase following a loss-of-coolant accident (LOCA).

In its capacity as the low-head portion of the ECCS, the RHRS provides long-term recirculation capability for core cooling following the injection phase of the LOCA. This function is accomplished by aligning the RHRS to take fluid from the containment sump, cool it by circulation through the residual heat exchangers, and supply it to the core directly as well as via the centrifugal charging pumps and safety injection pumps.

The use of the RHRS as part of the ECCS is more completely described in section 6.3.

The RHR pumps, in order to perform their ECCS function, are interlocked to start automatically on receipt of a safety injection signal (section 6.3).

The RHRS suction isolation valves in each inlet line from the RCS are separately interlocked to prevent both of them from being opened when RCS pressure is greater than approximately 365 psig, and the valves have a control room alarm, which alerts the operators if one or both of the valves is not fully closed and the RCS pressure exceeds 420 psig. These interlocks are described in more detail in paragraph 5.4.7.2.4 and subsection 7.6.2.

The RHRS suction isolation valves are also interlocked to prevent their being opened unless the isolation valves in the following lines are closed:

- A. Recirculation lines from the residual heat exchanger outlets to the suctions of the safety injection pumps and centrifugal charging pumps.
- B. RHR pump suction line from the refueling water storage tank.
- C. RHR pump suction line from the containment sump.

The motor-operated valves in the RHRS miniflow bypass lines are interlocked to open when the RHRS pump discharge flow is less than the open setpoint (824 gpm at 350 °F, 780 gpm at 100 °F) and to close when the flow exceeds the closed setpoint (1944 gpm at 350 °F, 1841 gpm at 100 °F).

5.4.7.2.2 Equipment and Component Descriptions

The materials used to fabricate RHRS components are in accordance with the applicable code requirements. All parts of components in contact with borated water are fabricated or clad with austenitic stainless steel or equivalent corrosion resistant material. Component parameters are given in table 5.4.7-2.

5.4.7.2.2.1 <u>Residual Heat Removal Pumps</u>. Two pumps are installed in the RHRS. The pumps are sized to deliver reactor coolant flow through the RHR heat exchangers to meet the plant cooldown requirements. The use of two separate RHR trains ensures that cooling capacity is only partially lost should one pump become inoperative.

The RHR pumps are protected from overheating and loss of suction flow by miniflow bypass lines that ensure flow to the pump suction. A valve located in each miniflow line is regulated by a signal from the flow transmitters located in each pump discharge header. The control valves open when the residual pump discharge flow is less than the open setpoint (824 gpm at 350 $^{\circ}$ F, 780 gpm at 100 $^{\circ}$ F) and close when the flow exceeds the closed setpoint (1944 gpm at 350 $^{\circ}$ F, 1841 gpm at 100 $^{\circ}$ F).

Although both valves could be closed by operator error, the design of the miniflow system would preclude any pump damage. The miniflow bypass valve is a fast operating motor-operated gate valve which will open or close in 10 s or less. The residual heat removal pump can operate safely without damage during this period with no flow.

A pressure sensor in each pump discharge header provides a signal for an indicator in the control room. A high-pressure alarm is also actuated by the pressure sensor.

The two pumps are vertical, centrifugal units with mechanical seals on the shafts. All pump surfaces in contact with reactor coolant are austenitic stainless steel or equivalent corrosion resistant material.

The RHR pumps also function as the low-head safety injection pumps in the ECCS (section 6.3).

A pump performance curve is provided in figure 5.4.7-2.

5.4.7.2.2.2 <u>Residual Heat Exchangers</u>. Two residual heat exchangers are installed in the system. The heat exchanger design is based on heat load and temperature differences between reactor coolant and CCW existing 20 h after reactor shutdown when the temperature difference between the two systems is small.

The installation of two heat exchangers in separate and independent RHR trains ensures that the heat-removal capacity of the system is only partially lost if one train becomes inoperative.

The residual heat exchangers are of the shell and U-tube type. Reactor coolant circulates through the tubes, while CCW circulates through the shell. The tubes are welded to the tube sheet to prevent leakage of reactor coolant.

The residual heat exchangers also function as part of the ECCS (section 6.3).

5.4.7.2.2.3 <u>RHRS Valves</u>. Valves that perform a modulating function are equipped with two sets of packing and an intermediate leakoff connection that discharges to a local equipment drain.

Manual and motor-operated valves have backseats to facilitate repacking and to limit stem leakage when the valves are open. Leakage connections are provided where required by valve size and fluid conditions. The RHR discharge cross-connect valves are provided with bonnet vents to the RHR pump side of the valves. RHR loop suction valves 1/2HV8701A, 1/2HV8702A, 1/2HV8702B, and 1HV8701B are provided with bonnet vents to the upstream side of the valves. The RHR loop suction valve 2HV8701B bonnet vent to the upstream side of the valve has been removed.

5.4.7.2.3 System Operation

5.4.7.2.3.1 <u>Plant Startup</u>. Generally, while in the cold shutdown condition, decay heat from the reactor core is being removed by the RHRS. The number of pumps and heat exchangers in service depends upon the heat load at the time.

At the beginning of plant startup, at least one RHR pump is operating, and a portion of the discharge flow may be directed to the CVCS.

This arrangement augments RCS pressure control during startup. When the reactor coolant pumps are started, the RHR pump is stopped. The thermal input of the reactor coolant pumps heats the reactor coolant inventory. Once the pressurizer steam bubble formation is complete, the RHRS is isolated from the RCS and aligned for operation as part of the ECCS.

- 5.4.7.2.3.2 <u>Power Generation and Hot Standby Operation</u>. During power generation and hot standby operation, the RHRS is not in service but is aligned for operation as part of the ECCS.
- 5.4.7.2.3.3 <u>Plant Shutdown</u>. Plant shutdown is defined as the operation that brings the plant from no-load temperature and pressure to cold conditions.
- 5.4.7.2.3.4 <u>Normal Cold Shutdown</u>. The initial phase of plant shutdown is accomplished by transferring heat from the RCS to the steam and power conversion system through the use of the steam generators.

When the reactor coolant temperature and pressure are reduced to approximately 350°F and 365 psig, approximately 2 to 4 hours after reactor shutdown, the second phase of cooldown starts with the RHRS being placed in operation.

Startup of the RHRS includes a warmup period, during which time reactor coolant flow through the heat exchangers is limited to minimize thermal shock. The rate of heat removal from the reactor coolant is manually controlled by regulating the coolant flow through the residual heat exchangers. By adjusting these control valves downstream of the residual heat exchangers, the mixed mean temperature of the return flows is controlled. Coincident with the manual adjustment of flow through the heat exchangers, each heat exchanger bypass valve is automatically regulated to give the required total flow.

The reactor cooldown rate is limited by RCS equipment cooling rates based on allowable stress limits and the operating temperature limits of the CCW system. As the reactor coolant temperature decreases, the reactor coolant flow through the residual heat exchangers is increased by adjusting the control valve in each heat exchanger's tube side outlet line.

Should both RHR heat exchanger outlet and bypass flow control valves fail simultaneously (i.e., loss of instrument air), then the maximum cooldown rate may be increased. The maximum cooldown rate depends on many factors, including the time of failure, the RHR flowrate, the CCW flowrates and temperatures, and other heat loads on the CCW system. One of the key factors is the RCS water temperature, since the cooldown rate depends upon the temperature difference between the RHR (RCS) flow and the CCW flow in the RHR heat exchanger. Even with the maximum flow through the RHR heat exchangers, it is typically impossible to maintain a cooldown rate as high as the technical specification design rate of 100°F/h when the RCS temperature is less than 250°F. The operator can significantly limit the cooldown rate by merely stopping one of the RHR pumps.

During plant shutdown, pressurizer steam bubble operation is maximized to control RCS pressure. When the RHRS is in operation, RCS control is augmented by regulating the charging flowrate and the rate of letdown from the RHRS to the CVCS.

After the reactor coolant pressure is reduced and the temperature is 140°F or lower, the RCS may be opened for refueling or maintenance.

A failure modes and effects analysis for normal cooldown operations is provided in table 5.4.7-3.

5.4.7.2.3.5 <u>Safety-Grade Cold Shutdown</u>. It is expected that the systems normally used for cold shutdown will be available anytime the operator chooses to perform a reactor cooldown. However, to ensure that the plant can be taken to cold shutdown at anytime, the safety-grade cold shutdown design enables the RCS to be taken from no-load temperature and pressure to cold conditions using only safety-grade systems, with only onsite or offsite power available, and assuming the most limiting single failure.

Should portions of normal shutdown systems be unavailable, the operator will maintain the plant in a hot standby condition while making those normal systems functional. Local manual actions are performed as described in table 5.4.7-4. Appropriate procedures are provided for the use of safety-grade backups contingent upon the inability to make normal systems available. The operator should use any of the normal systems that are available in combination with the safety-grade backups for the systems that cannot be made operable. The safety-grade provisions are to be used only upon the inability to make available the equipment normally used for the given function.

The safety-grade cold shutdown design enables the operator to maintain the plant in hot standby for approximately 4 h. Since it is assumed that the reactor coolant pumps are not available, circulation of the reactor coolant is provided by natural circulation with the reactor core as the heat source and the steam generators as the heat sink. Heat removal is accomplished via the steam generator power-operated relief valves and auxiliary feedwater system.

The boration of the RCS is initiated prior to cooling the RCS. The charging pumps are used to provide borated water to the RCS at a rate of approximately 0 to 50 gal/min. The borated water is delivered to the RCS via the safety-grade charging line or the high-head safety injection lines. Both flow paths have provisions for flow control. Reactor coolant pump seal injection is also maintained. To accommodate this addition to RCS inventory, continuous letdown is discharged from the reactor vessel head letdown line to the pressurizer relief tank.

During boration to cold shutdown concentration, the safety-grade cooldown is accomplished by increasing the steam dump from the steam generator power-operated relief valves to attain a rate of primary side cooling of approximately 35°F/h. In conjunction with this portion of the cooldown, the charging pumps are used to deliver borated water to makeup for primary contraction due to cooling. Makeup is also provided for the RCS inventory discharged when the reactor vessel head letdown path is periodically cycled to provide head cooling. Upon approaching the end of this phase of cooldown, the RCS is depressurized by venting or letdown through the reactor vessel head vent valves to the pressurizer relief tank.

To ensure that the accumulators do not repressurize the RCS, the accumulator discharge valves are closed prior to the RCS pressure dropping below the accumulator discharge pressure. Additionally, each accumulator is provided with two Class 1E solenoid-operated valves in parallel to ensure that the accumulator may be vented should it fail to be isolated from the RCS.

When the reactor coolant temperature and pressure are reduced to approximately 350°F and 365 psig, respectively, and the RCS borated to cold shutdown concentration, the second phase of cooldown starts with the RHRS being placed in operation.

As safety-grade cooldown continues, the reactor vessel head letdown line is periodically opened to increase head cooling and to accommodate any additional input to the RCS, such as reactor

coolant pump seal injection. Since loss of nonsafety-grade equipment results in a loss of the air supply to the flow control valves that are normally used to limit the initial RHRS cooldown rate, the operator may choose to use only one of the RHR subsystems. Should a single failure, such as that of an RHRS component or of an emergency power train (when only onsite power is available), limit operation to one of the RHR subsystems, the operator would open the series isolation valves in the suction of only the operable RHR subsystem. In this case, the operator would also close the cross-connect isolation valves between the subsystems. RHR would continue under these conditions until the redundant subsystem could be made available.

A failure modes and effects analysis for safety-grade cold shutdown operations is provided in table 5.4.7-4.

5.4.7.2.3.6 <u>Refueling</u>. The RHRS is utilized during refueling to transfer borated water from the refueling water storage tank to the refueling cavity. During this operation, one RHR train is selected for fill, the isolation valves in the suction lines from the RCS are closed, the isolation valves from the refueling water storage tank are opened and the RHR pump may be started or the refueling water storage tank (RWST) water is allowed to gravity drain.

The refueling cavity is prepared for flooding and the vessel head is removed to its storage pedestal using the containment polar crane. The refueling water is then transferred into the reactor vessel through the RHRS hot-leg or cold-leg return lines and into the refueling cavity through the open reactor vessel. After the water level reaches the normal refueling level, the RHR pump is stopped, the refueling water storage tank supply valves are closed, and the suction isolation valves from the RCS are opened.

During refueling, the RHRS is maintained in service with the number of pumps and heat exchangers in operation as required by the heat load. Also during refueling, an RHR pump not being applied for shutdown cooling can be used to satisfy the Technical Requirements Manual requirement that a pump be available for boration when the refueling cavity water level is ≥ 23 feet above the flange. The flowpath associated with this application of the RHR pump is the pump taking suction from the RWST and injecting into the RCS cold legs.

Following refueling, the RHRS is used to drain the refueling cavity to the top of the reactor vessel flange by pumping water from the RCS to the refueling water storage tank. The vessel head is then replaced and the normal RHRS flow path reestablished. The remainder of the water is removed from the refueling canal via a drain connection in the bottom of the canal.

An alternative methodology may be used for raising the vessel head. In this method, the reactor vessel head is lifted slightly. Then the cavity is filled as discussed above, while reactor vessel head is gradually raised as the water level in the refueling cavity increases. The filling of the cavity is also terminated as discussed above. Following refueling, placement of the head on the vessel is accomplished by maintaining the vessel head just above the water in the cavity, as the level lowers while the refueling water is being pumped from the cavity to the RWST by the RHR pumps.

5.4.7.2.3.7 <u>Mid-loop and Drain Down Operations.</u> The RHR system is used to provide core cooling when the RCS must be partially drained to allow maintenance or inspection of the reactor head, steam generators, or reactor coolant pump seals.

The level in the primary system is lowered to near the mid-line of the hot and cold legs. At this water level the air/water interface is at close proximity to the RHR suction nozzles located on the hot legs of loops 1 and 4, and care must be taken to avoid air entrainment into the RHR

pump suction. Air ingestion by an RHR pump can cause loss of pump function, creating the potential for loss of residual heat removal.

Instrumentation has been provided to assist the operator in safely maintaining adequate level in the RCS hot legs during mid-loop and drain down operations. This instrumentation is shown in drawings 1X4DB111, 2X4DB111, 1X4DB112, 2X4DB112, and 1X4DB113. Instrumentation has also been provided to assist the operator in quickly identifying air ingestion in the RHR pumps.

Two differential pressure transmitters are connected to the RCS to provide independent level indications in the main control room. One transmitter is connected to the RCS loop 1 hot leg and provides narrow range indication. This narrow range indicator spans from the bottom of the hot leg upward 8 feet. This instrument loop also provides annunciation of the low hot leg level. The other transmitter is connected to the RCS loop 4 hot leg and provides wide range indication. This wide range indicator spans from the bottom of the hot leg upward 20 feet. The instrument loops are powered from separate breakers to maximize the availability of the indication.

Local RCS level indication is available via a sight glass located in the containment building. The sight glass spans from the top of the pressurizer heaters down to just below the bottom of the RCS hot legs. The piping for this sight glass is connected to the RCS as required during modes 5 and 6. The tubing used for the sight glass is vacuum resistant.

Current transducers monitor the 4,160 V power feeders to each RHR pump. The output of these transducers is routed to the plant computer. Historical traces of the pump motor current can be obtained at any plant computer terminal. The logic associated with modes 5 and 6 core cooling critical safety function status trees provides a visual alarm at the plant computer safety parameter display system (SPDS) terminal in the main control room, if the motor current becomes unstable. The critical safety function status trees alarm the control room annunciator when any adverse condition occurs.

5.4.7.2.4 Control

Each inlet line to the RHRS is equipped with a relief valve to prevent RHRS overpressurization during plant startup, shutdown, and cold shutdown decay heat-removal operation. Each valve has a relief capacity of 900 gal/min at a set pressure of 450 psig. An analysis has been conducted to confirm the capability of the RHRS relief valve to prevent overpressurization in the RHRS. All credible events were examined for their potential to overpressurize the RHRS. These events included normal operating conditions, infrequent transients, and abnormal occurrences. The analysis confirmed that one relief valve has the capability to maintain the RHRS maximum pressure within code limits. The above capacities of the RHRS suction line relief valves are adequate to provide relief protection necessary for the RHRS and the RCS as part of the cold overpressure mitigating system. For a discussion of the cold overpressure mitigating system and the overpressure events examined, refer to WCAP-10529.

Each discharge line from the RHRS to the RCS is equipped with a pressure relief valve to relieve the maximum possible back-leakage through the valves separating the RHRS from the RCS. Each valve has a relief flow capacity of 20 gal/min at a set pressure of 600 psig. These relief valves are located in the ECCS (drawing 1X4DB121).

The fluid discharged by the suction-side relief valves is collected in the pressurizer relief tank. The fluid discharged by the discharge side relief valves is collected in the recycle holdup tank.

The design of the RHRS includes two motor-operated gate isolation valves in series on each inlet line between the high-pressure RCS and the lower pressure RHRS. They are closed during normal operation and are opened only for RHR during a plant shutdown after the RCS

pressure is reduced to approximately 365 psig or lower and RCS temperature is reduced to approximately 350°F. During a plant startup, the inlet isolation valves are shut after drawing a bubble in the pressurizer and prior to increasing RCS pressure above 425 psig. These isolation valves are provided with independent and diverse "prevent-open" interlocks and main annunciator alarms which are designed to prevent possible exposure of the RHRS to normal RCS operating pressure. The two inlet isolation valves in each RHR subsystem are independently interlocked with diverse pressure signals to prevent their being opened whenever the RCS pressure is greater than approximately 365 psig. Additionally, during plant startup, the valves are interlocked such that they alarm in the main control room when RCS pressure increases to 420 psig to alert the operators that one or both of the valves is not fully closed.

The use of two independently powered motor-operated valves in each of the two inlet lines, with an independently powered pressure transmitter for each valve, along with two independent and diverse interlock signals for each function, ensures a design which meets applicable single-failure criteria. These protective interlock designs, in combination with plant operating procedures, provide the means of accomplishing the protective function. For further information on the instrumentation and control features, see subsection 7.6.2.

The RHR inlet isolation valves are provided with control switches with integral red-green position indicator lights on the main control board and an alarm window on a main control board annunciator panel.

Isolation of the low-pressure RHRS from the high-pressure RCS is provided on the discharge side by two check valves in series. These check valves are located in the ECCS, and their testing is described in paragraph 6.3.4.2.

5.4.7.2.5 Applicable Codes and Classifications

The entire RHRS is designed as Nuclear Safety Class 2 with the exception of the suction isolation valves, which are Safety Class 1. Component codes and classifications are given in section 3.2.

5.4.7.2.6 System Reliability Considerations

General Design Criterion 34 requires that a system to remove residual heat be provided. The safety function of this system is to transfer fission product decay heat and other residual heat from the core at a rate sufficient to prevent fuel or pressure boundary design limits from being exceeded. Safety-grade systems are provided in the plant design, both nuclear steam supply system scope and balance of plant scope, to perform this function. The safety-grade systems that perform this function for all plant conditions except a LOCA are the RCS and steam generators, which operate in conjunction with the auxiliary feedwater system and the steam generator safety and power-operated relief valves, and the RHRS, which operates in conjunction with the CCW system and the nuclear service cooling water (NSCW) system. For LOCA conditions, the safety-grade system which performs the function of removing residual heat from the reactor core is the ECCS, which operates in conjunction with the CCW system and the NSCW system.

The auxiliary feedwater system, along with the steam generator safety and power-operated relief valves, provides a completely separate, independent, and diverse means of performing the safety function of removing residual heat, which is normally performed by the RHRS when RCS temperature is less than 350°F. The auxiliary feedwater system is capable of performing this function for an extended period of time following plant shutdown.

The RHRS is provided with two RHR pumps and two residual heat exchangers arranged in two separate, independent flow paths. To ensure reliability, each RHR pump is connected to a different vital bus. Each RHR train is isolated from the RCS on the suction side by two motor-operated valves in series, with each valve receiving power via a separate motor control center and from a different vital bus. The suction isolation valves prevent exposure of the RHRS to the normal operating pressure of the RCS (paragraph 5.4.7.2.4).

The RHRS operation for normal conditions, even with a major failure, is accomplished completely from the control room except as described in paragraph 5.4.7.2.7. The redundancy in the RHRS design provides the system with the capability to maintain its cooling function, even with a major single failure such as failure of a pump, valve, or heat exchanger, since the redundant train can be used for continued heat removal.

Status-indicating lights are provided at the control board for the RHR pump, the RHRS suction isolation valves, and the miniflow isolation valves.

The major portion of the RHRS is located in the auxiliary building. Leakages resulting from a passive failure of the RHRS piping are collected by the floor drain system. See subsection 9.3.3 for a discussion of this system and the alarms and instrumentation provided to detect any radioactive leaks, should they occur.

5.4.7.2.7 Manual Actions

The RHRS is designed to be fully operable from the control room for normal operation except for restoring power to the suction isolation valves, and for restoring air to the heat exchanger flow control valves prior to RHR initiation. Manual operations required of the control room operator are restoring power to and opening the suction isolation valves, restoring air to and positioning the flow control valves downstream of the residual heat exchangers, and starting the RHR pumps. The power lockout of the suction isolation valves is discussed in subsection 7.6.2. The air is isolated from the RHR heat exchanger flow control valves by closing the air supply isolation valve. During normal cooldown, there is adequate time and accessibility to perform these actions. Refer to paragraph 6.3.2.8 for the manual actions required during ECCS operations.

5.4.7.3 <u>Performance Evaluation</u>

The performance of the RHRS in reducing reactor coolant temperature is evaluated through the use of heat balance calculations on the RCS, RHRS, and CCW system at stepped intervals following the initiation of removal operation. Heat removal through the RHR and component cooling heat exchangers is calculated at each interval by use of standard water-to-water heat exchanger performance correlations; the resultant fluid temperatures for the RCS, RHRS, and CCW system are calculated and used as input to the next interval's heat balance calculation.

Assumptions utilized in the series of heat balance calculations describing plant RHRS cooldown are as follows:

- A. RHR operation is initiated 4 h after reactor shutdown. (a)
- B. RHR operation begins at a reactor coolant temperature of 350°F.

^(a) This value is used in design bases criteria and analyses to support the plant maximum cold shutdown condition. The RHR system and support systems have been reviewed and evaluated to document the acceptability of an operational cooling rate up to 100°F/h.

- C. Thermal equilibrium is maintained throughout the RCS during the cooldown (i.e., one reactor coolant pump in operation whenever the reactor coolant temperature is above 160°F).
- D. CCW temperature during cooldown is limited to a maximum of 120°F.
- E. RCS cooldown rates of 50°F/h are not exceeded. (b)

Cooldown curves calculated using this method are provided in figure 5.4.7-3.

5.4.7.4 <u>Preoperational Testing</u>

Preoperational testing of the RHRS is addressed in chapter 14.

5.4.7.5 Reliability Tests and Inspections

As the RHRS functions as part of the ECCS, periodic tests and inspections are conducted in conjunction with those conducted on ECCS components. (See paragraph 6.3.4.2 for a discussion of the tests and inspections. See the Technical Specifications for the selection of test frequency, acceptability of testing, and measured parameters. A description of the inservice inspection program is included in section 6.6.)

5.4.8 REACTOR WATER CLEANUP SYSTEM

This subsection is not applicable to VEGP.

5.4.9 MAIN STEAM LINE AND FEEDWATER PIPING

The main steam and feedwater systems of VEGP are not part of the reactor coolant system. The main steam system, including the main steam piping, is described in section 10.3. The feedwater system, including piping, is described in subsection 10.4.7.

5.4.10 PRESSURIZER

5.4.10.1 <u>Design Bases</u>

The pressurizer provides a point in the reactor coolant system (RCS) where liquid and vapor are maintained in equilibrium under saturated conditions for control of pressure of the RCS during steady-state operations and transients.

The volume of the pressurizer is equal to, or greater than, the minimum volume of steam, water, or total of the two which satisfies all of the following requirements:

- A. The combined saturated water volume and steam expansion volume is sufficient to provide the desired pressure response to system volume changes.
- B. The water volume is sufficient to prevent the heaters from being uncovered during a step-load increase of 10% at full power.

- C. The steam volume is large enough to accommodate the surge resulting from a 50% reduction of full load with automatic reactor control and a 40% steam dump without the water level reaching the high level reactor trip point.
- D. The steam volume is large enough to prevent water relief through the safety valves following a loss of load with the high water level initiating a reactor trip, without reactor control or steam dump.
- E. The pressurizer will not empty following reactor trip and turbine trip.
- F. A low pressurizer pressure safety injection signal will not be activated because of a reactor trip and turbine trip.

The surge line is sized to maintain the pressure drop between the RCS and the safety valves within allowable limits during a design discharge flow from the safety valves.

The surge line is designed to withstand the thermal stresses resulting from volume surges occurring during operation. In accordance with TMI Action Item II.K.3.9, the derivative action setting of the proportional integral derivative controller has been set to zero to prevent spurious power-operated relief valves (PORV) operation.

5.4.10.2 <u>Design Description</u>

5.4.10.2.1 Pressurizer and Surge Line

The pressurizer is a vertical, cylindrical vessel having hemispherical top and bottom heads constructed of carbon steel, with austenitic stainless steel cladding on all internal surfaces exposed to the reactor coolant. The surge line is constructed of stainless steel.

The general configuration of the pressurizer is shown in figure 5.4.10-1. The design data for the pressurizer are given in table 5.4.10-1. A comparison of the design basis operating conditions determined that the conditions originally qualified for the pressurizer will envelop the operation at a power level of 3653 MWt. Codes and material requirements are provided in section 5.2.

The pressurizer surge line connects the pressurizer to one reactor coolant hot leg, thus enabling continuous coolant volume and pressure adjustments between the RCS and the pressurizer.

The surge line nozzle and electric heaters are located in the bottom head of the pressurizer. The heaters are designed to be removable for maintenance or replacement.

The pressurizer surge line nozzle diameter is given in table 5.4.10-1, and the pressurizer surge line size is shown in drawing 1X4DB112.

A retaining screen is located above the nozzle to prevent passage of any foreign matter from the pressurizer to the RCS. Baffles in the lower section of the pressurizer prevent an insurge of cold water from flowing directly to the steam/water interface and also assist in mixing.

The spray line nozzles and the relief and safety valve connections are located in the top head of the pressurizer vessel. Spray flow is modulated by automatically controlled air-operated valves. The spray valves also can be operated manually from the control room.

A small continuous spray flow is provided through a manual bypass valve around each poweroperated spray valve to minimize the boron concentration difference between the pressurizer liquid and the reactor coolant and to prevent excessive cooling of the spray piping. During an outsurge of water from the pressurizer, flashing of water to steam and generation of steam by automatic actuation of the heaters keep the pressure above the low-pressure reactor trip setpoint. During an insurge from the RCS, the spray system, which is fed from two cold legs, condenses steam in the pressurizer to prevent the pressurizer pressure from reaching the setpoint of the PORVs. The heaters are energized on high water level during insurge to heat the subcooled surge water that enters the pressurizer from the reactor coolant loop. The skirt-type support is attached to the lower head and extends for a full 360° around the vessel. The lower part of the skirt terminates in a bolting flange with bolt holes for securing the vessel to its foundation. The support is provided with ventilation holes to ensure free convection of ambient cooling air past the heater and connector ends.

Material specifications are provided in table 5.2.3-1 for the pressurizer and the surge line. Design transients for the components of the RCS are discussed in subsection 3.9.N.1.

5.4.10.2.2 Pressurizer Spray and Relief Line Instrumentation

Refer to chapter 7 for details of the instrumentation associated with pressurizer pressure, level, and temperature.

Temperatures in the spray lines from the cold legs of two loops are measured and indicated. Alarms from these signals are actuated to warn the operator of low spray water temperature or to indicate insufficient flow in the spray lines.

Temperatures in the pressurizer safety and relief valves discharge lines are measured and indicated. An increase in a discharge line temperature is an indication of leakage or relief through the associated valve. High temperature alarms are initiated if the leakage is abnormal.

5.4.10.3 Design Evaluation

5.4.10.3.1 System Pressure Control

The RCS pressure is controlled by the pressurizer whenever a steam volume is present in the pressurizer.

A design basis safety limit has been set such that the RCS pressure does not exceed the maximum transient value allowed under the American Society of Mechanical Engineers (ASME) Code, Section III. Evaluation of plant conditions of operation considered for design indicates that this safety limit is not reached.

During startup and shutdown, the rate of temperature change in the RCS is controlled by the operator. Heatup rate is controlled by energy input from the reactor coolant pumps and by the pressurizer electrical heating capacity.

The pressurizer heaters are powered from four electrical panels: one panel for each heater group. Two groups of heaters can be administratively loaded onto the non-Class 1E emergency buses (drawings 1X3D-AA-A01A and 2X3D-AA-A01A). The Class 1E 4160-V breakers supplying the non-Class 1E buses are automatically opened upon a safety injection signal. The non-Class 1E buses can be manually reenergized under administrative procedure. Actions associated with these two groups of heaters can be controlled from either the control room or the shutdown panels. Engineered safety features loads need not be shed to manually load a pressurizer heater group.

Pressurizer heater controls are described in paragraph 7.7.1.D.

Analysis performed for a four-loop plant with an 1800-ft³ pressurizer similar to VEGP indicates that a heater capacity of 150 kW is adequate to maintain subcooled conditions in the RCS during natural circulation. Each VEGP heater panel has a heater capacity of 483 kW. Furthermore, the analysis demonstrates that the RCS sensible heat capacity is such that adequate subcooling can be maintained in the RCS for 4 h without heat input from the pressurizer heaters. Thus the recommendations of Action Item II.E.3.1 of NUREG-0737 are satisfied.

When the pressurizer is filled with water, i.e., during initial system heatup or near the end of the second phase of plant cooldown, RCS pressure is controlled by the letdown flowrate via the residual heat removal system.

5.4.10.3.2 Pressurizer Level Control

The normal operating water volume at full-load conditions is approximately 60% of the free internal vessel volume. Under part-load conditions the water volume in the pressurizer is reduced proportionally with reductions in plant load to approximately 25% of the free internal vessel volume at the zero-power condition.

5.4.10.3.3 Pressure Setpoints

The RCS design and operating pressure, together with the safety valve setpoints, PORV setpoints, pressurizer spray valve setpoints, and the protection system pressure setpoints, are listed in table 5.4.10-2. The design pressure allows for operating transient pressure changes. The selected design margin considers core thermal lag, coolant transport times and pressure drops, instrumentation and control response characteristics, and system relief valve characteristics. The low pressurizer pressure reactor trip does not require a coincident low water level signal. This is in accordance with the recommendations of Action Item II.K.1.17 of NUREG-0660.

Temperature changes which may affect the relief valve setpoints have been considered. Normal ambient air temperature variations have no significant effects. However, cold valves relieving hot fluid may show reduced setpoints; therefore, this has been considered in the design of such valves.

5.4.10.3.4 Pressurizer Spray

Two separate automatically controlled spray valves with remote manual overrides are used to initiate pressurizer spray. In parallel with each spray valve is a manual throttle valve which permits a small continuous flow through both spray lines to reduce thermal stresses and thermal shock when the spray valves open and to help maintain uniform water chemistry and temperature in the pressurizer. Temperature sensors with low alarms are provided in each spray line to alert the operator to insufficient bypass flow. The layout of the common spray line piping routed to the pressurizer forms a water seal which prevents the steam buildup back to the control valves. The design spray rate is selected to prevent the pressurizer pressure from reaching the operating setpoint of the PORVs during a step reduction in power level of 10% of full load.

The pressurizer spray lines and valves are large enough to provide the required spray flowrate under the driving force of the differential pressure between the surge line connection in the hot leg and the spray line connection in the cold leg. The spray line inlet connections extend into the cold leg piping in the form of a scoop in order to utilize the velocity head of the reactor coolant loop flow to add to the spray driving force. The spray valves and spray line connections are arranged so that the spray operates, although at a reduced capacity when one reactor coolant pump is not operating. The spray line also assists in equalizing the boron concentration between the reactor coolant loops and the pressurizer.

A flowpath from the chemical and volume control system to the pressurizer spray line is also provided. This path provides auxiliary spray to the vapor space of the pressurizer during cooldown when the reactor coolant pumps are not operating. The pressurizer spray connection and the spray piping are designed to withstand the thermal stresses resulting from the introduction of cold spray water.

5.4.10.4 Tests and Inspections

The pressurizer is designed and constructed in accordance with the ASME Code, Section III.

To implement the requirements of the ASME Code, Section XI, the following welds are designed and constructed to present a smooth transition surface between the parent metal and the weld metal. The weld surface is ground smooth for ultrasonic inspection.

- Support skirt to the pressurizer lower head.
- Surge nozzle to the lower head.
- Safety, relief, and spray nozzles to the upper head.
- Nozzle to safe end attachment welds.
- All girth and longitudinal full-penetration welds.
- Manway attachment welds.

The liner within the safe end nozzle region extends beyond the weld region to maintain a uniform geometry for ultrasonic inspection.

Peripheral support rings are furnished for the removable insulation modules.

The pressurizer quality assurance program is given in table 5.4.10-3.

Tests and inspections performed under the following license renewal aging management programs are credited as applicable to the pressurizer and its subcomponents:

- Bolting Integrity Program (see subsection 19.2.2).
- Boric Acid Corrosion Control Program (see subsection 19.2.3).
- Inservice Inspection Program (see subsection 19.2.13).
- Nickel Alloy Management Program for Nonreactor Vessel Closure Head Penetration Locations (see subsection 19.2.14).

- Water Chemistry Control Program (see subsection 19.2.28).
- Fatigue Monitoring Program (see subsection 19.3.2).

5.4.11 PRESSURIZER RELIEF DISCHARGE SYSTEM

5.4.11.1 Design Bases

The pressurizer relief discharge system collects, cools, and directs the steam and water discharged from various safety and relief valves in the containment for processing. The system consists of the pressurizer relief tank (PRT), the pressurizer safety and relief valve discharge piping, the relief tank internal spray header and associated piping, the tank nitrogen supply, and the drain to the liquid waste processing system.

The system design, including the PRT design volume, is based on the requirement to condense and cool a discharge of steam equivalent to 110% of the full-power pressurizer steam volume, without exceeding a pressure/temperature condition of 50 psig/200°F in the PRT. These values are well below the PRT design conditions of 100 psig and 340°F. Additional design data for the tanks are given in table 5.4.11-1.

The minimum volume of water in the PRT is determined by the energy content of the steam to be condensed and cooled, by the assumed initial temperature of the water, and by the desired final temperature of the water volume. The initial water temperature is assumed to be 120°F, which corresponds to the design maximum expected containment temperature for normal conditions. Provision is made to permit cooling of the water in the tank should the water temperature rise above 120°F during plant operation. The design final temperature, following a design discharge to the tank, is 200°F, which allows the contents of the tank to be drained directly to the liquid waste processing system without cooling.

The PRT saddle supports and anchor bolt arrangement are designed to withstand the loadings resulting from the vessel seismic, static, and nozzle loadings.

The pressurizer safety and relief valve piping and support arrangement is designed such that the effect of thrust forces on the piping system from valve operations is minimized. The piping analysis is discussed in section 3.9.N.

The design and location of the PRT rupture disks are such that they do not pose a missile threat to any safety-related equipment.

5.4.11.2 System Description

The piping and instrumentation diagram for the pressurizer relief discharge system is given in drawing 1X4DB112.

The steam and reactor grade water discharged from the various safety and relief valves inside the containment is routed to the PRT. Table 5.4.11-2 provides an itemized list of the discharges to the tank, together with references to the corresponding piping and instrumentation diagrams.

The pressurizer safety and relief valve piping and support arrangement in figure 5.4.11-2 shows the valve discharge piping, as well as the piping upstream of the safety and relief valves. The

piping upstream of the valves, which is not considered part of the pressurizer relief discharge system, includes the following:

- A. Three lines with loop seal arrangements connecting the pressurizer nozzles to the three safety valves.
- B. A line from the pressurizer relief nozzle branching to the two power-operated relief valves (PORVs), which have individual water seals and motor-operated isolation valves.

The pressurizer safety and relief valve discharge piping consists of:

- A. A common piping manifold (supported over the top of the pressurizer) into which the safety and relief valves discharge.
- B. Safety valve discharge lines to the manifold.
- C. Relief valve discharge lines to the manifold.
- D. A manifold downcomer discharge pipe.
- E. Piping to the PRT.

The main support structure for the safety and relief valve piping consists of four column members equally spaced around the common manifold coupled to the valve support brackets on the pressurizer. No welding to the pressurizer is required. To increase the natural frequency of the system, auxiliary crossmembers are provided from the common manifold to the main support columns. The safety valves are provided with a bottom saddle type support coupled to the auxiliary crossmembers. The relief valves are positioned above the manifold, and the relief valve lines are supported at various points along the manifold.

The pressurizer safety and relief valve piping is constructed of austenitic stainless steel. Design data for the pressurizer safety and relief valve piping are given in table 5.4.3-1.

The piping upstream of the safety and relief valves is part of the reactor coolant system (RCS) and is designed and fabricated in accordance with American Society of Mechanical Engineers (ASME) Code, Section III, Class 1 requirements. The piping between these valves and the downcomer tee connection is nonnuclear safety related but is designed and fabricated to ASME Code, Section III, Class 2, to the extent practical. The support structure for the piping from the pressurizer to the downcomer tee connection is designed and fabricated to ASME Code, Section III, Subsection NF.

The piping from the pressurizer to the PRT is designed to Seismic Category 1 requirements. The principal design codes are indicated in table 3.2.2-1.

The general configuration of the PRT is shown in figure 5.4.11-1. The tank is a horizontal, cylindrical vessel with elliptical dished heads. The vessel is constructed of austenitic stainless steel and is overpressure protected by means of two safety heads with stainless steel rupture discs. Also shown in figure 5.4.11-1 are the flanged connection for the pressurizer safety and relief valve discharge line, the spray water inlet, the bottom drain connection, the gas vent connection, and the vessel supports. Although the tank is classified as nonnuclear safety related, it is designed and fabricated to Section III, Division 1, Class 3 of the ASME Code.

The tank normally contains water and a predominantly nitrogen atmosphere. In order to obtain effective condensing and cooling of the discharged steam, the tank is installed horizontally so that the steam can be discharged through a sparger pipe located near the bottom, under the water level. The sparger holes are designed to ensure good mixing of the discharged steam with the water initially in the tank.

A nitrogen gas blanket is used to control the atmosphere in the tank and to allow room for the expansion of the original water, plus the condensed steam discharge. The tank gas volume is sized such that the pressure following a design basis steam discharge does not exceed 50 psig, assuming an initial pressure of 3 psig. This pressure is low enough to prevent opening of the rupture discs. Provisions are made to permit the gas in the tank to be periodically analyzed to determine the concentration of hydrogen and/or oxygen.

The internal spray and bottom drain on the PRT function to cool the water when the temperature exceeds 120°F, as in the case following a steam discharge. The contents are cooled by a feed-and-bleed process, with cold reactor makeup water entering the tank through the spray water inlet and the warm mixture draining to the reactor coolant drain tank (RCDT). The contents may also be cooled by recirculation through the RCDT heat exchanger of the liquid waste processing system.

5.4.11.3 Safety Evaluation

The pressurizer relief discharge system does not constitute part of the reactor coolant pressure boundary in accordance with 10 CFR 50.2, since all of its components are downstream of the RCS safety and relief valves; thus, General Design Criteria 14 and 15 are not applicable. Furthermore, complete failure of the auxiliary systems serving the PRT will not impair the capability for safe plant shutdown.

The design of the system piping layout and piping restraints is consistent with the hazards protection requirements discussed in section 3.6 and appendix 3F. The safety and relief valve discharge piping is restrained so that the integrity and operability of the valves are maintained in the event of a rupture. Regulatory Guide 1.67 is not applicable since the system is not an open discharge system.

The pressurizer relief discharge system is capable of handling the design discharge of steam without exceeding the design pressure and temperature. The volume of nitrogen in the PRT is that required to limit the maximum pressure accompanying the design basis discharge to 50 psig, half the design pressure of the tank. The volume of water in the PRT is capable of absorbing the heat from the assumed discharge while maintaining the water temperature below 200°F.

If a discharge results in a pressure that exceeds the design, the rupture discs on the tank would pass the discharge through the tank to the containment. The rupture discs on the relief tank have a relief capacity equal to or greater than the combined capacity of the pressurizer safety valves. The tank and rupture discs holders are also designed for full vacuum to prevent tank collapse, if the contents cool following a discharge without nitrogen being added.

The discharge piping from the pressurizer safety and relief valves to the PRT is sufficiently large to prevent backpressure at the safety valves from exceeding 20% of the setpoint pressure at full flow.

The recommendations of NUREG-0737, Action Items II.G.1 and II.K.3.1, are met as discussed below. The pressurizer is equipped with two Class 1E PORVs (solenoid operated) and two Class 1E PORV block valves (motor operated). The PORV and associated block valve on one line are supplied with control and motive power from train A, while the other PORV and associated block valve on the other line are powered from train B (drawings 1X4DB111, 2X4DB111, 1X4DB112, 2X4DB112, and 1X4DB113).

The PORV block valves 1HV-8000A and 1HV-8000B are powered from Class 1E 480-V buses. These buses are normally supplied from offsite power. In the event of a loss of offsite power,

these buses are automatically loaded onto the diesels (drawings 1X3D-AA-A01A and 2X3D-AA-A01A). PORVs 455A and 456A are Class 1E dc solenoid valves and are powered from redundant Class 1E 125-V dc trains A and B, respectively. The train assignment for power to the PORVs and block valves is based on:

- A. The ability to open one of the parallel pressurizer vent paths in conjunction with a single failure.
- B. The ability to close both parallel paths in conjunction with a single failure. (Capability to isolate both parallel paths in conjunction with a single failure is based upon the fact that the solenoid-operated PORVs are qualified, dc powered, and designed to fail closed.)

Pressurizer pressure is interlocked with the PORV block valves, which provides automatic closure of the block valves upon low pressurizer pressure.

5.4.11.4 Instrumentation Requirements

The following instrumentation is provided on the main control board:

- A. The PRT pressure transmitter provides a signal to an indicator. An alarm is provided to indicate high tank pressure.
- B. The PRT level transmitter supplies a signal to an indicator. High- and low-level alarms are also provided.
- C. The temperature of the water in the PRT is displayed by an indicator. An alarm actuated by high temperature informs the operator that cooling of the tank contents is required.
- D. The temperature of the safety and relief valve discharge lines is displayed by indicators. Alarms actuated by high temperature notify the operator of steam discharge due to either leakage or valve actuation.

5.4.11.5 Inspection and Testing Requirements

The nondestructive examinations performed during fabrication of the forged piping from the pressurizer to the downcomer tee connection are identified in table 5.4.11-3.

The PRT is subject to nondestructive and hydrostatic testing during construction and after installation in accordance with Section III, Division 1, Class 3 of the ASME Code.

The downcomer piping to the PRT is subject to nondestructive and hydrostatic testing during construction.

Periodic visual inspections and preventive maintenance are conducted on the system components according to normal industrial practice.

5.4.12 **VALVES**

5.4.12.1 <u>Design Bases</u>

As noted in section 5.2, all valves out to and including the second valve that is normally closed or capable of automatic or remote closure, larger than 3/4 in., are American Society of Mechanical Engineers (ASME) Code, Section III, Class 1 valves. Valves 3/4-in. or smaller in lines connected to the reactor coolant system (RCS) are Class 1, unless the Class 1 piping interface is provided with suitable flow limiting orificing, then the valves are Class 2.

For a check valve to qualify as part of the RCS, it must be located inside the containment system. When the second of two normally closed check valves is considered part of the RCS (as defined in section 5.1), means are provided to periodically assess backflow leakage of the first valve when closed.

To ensure that the valves will meet the design objectives, the materials of construction minimize corrosion/erosion and are compatible with the environment. Leakage is minimized to the extent practicable by design.

5.4.12.2 Design Description

All manual- and motor-operated valves of the RCS which are 3 in. and larger are provided with double-packed stuffing boxes and intermediate lantern ring leakoff connections or a reduced packing configuration with the valve stem leakoff line removed. Throttling control valves over 2 in. are provided with double-packed stuffing boxes and stem leakoff connections. In general, RCS leakoff connections are piped to a closed collection system. Leakage to the atmosphere is essentially zero for these valves.

Gate valves at the engineered safety features interface are wedge design and are essentially straight through. The wedges are flex-wedge or solid. All gate valves have backseats.

Globe valves are "T" and "Y" styles of outside screw and yoke construction.

Check valves are swing type for sizes 2 1/2 in. and larger. All check valves which contain radioactive fluid are stainless steel and do not have body penetrations other than the inlet, outlet, and bonnet. The check hinge is serviced through the bonnet. All operating parts are contained within the valve body. The disc has limited rotation to provide a change of seating surface and alignment after each valve opening.

5.4.12.3 Design Evaluation

The design requirements for Class 1 valves, as discussed in section 5.2, limit stresses to levels which ensure the structural integrity of the valves. In addition, the testing programs described in section 3.10.N demonstrate the ability of the valves to operate, as required, during anticipated and postulated plant conditions.

Reactor coolant chemistry parameters are specified in the design specifications to ensure the compatibility of valve construction materials with the reactor coolant. To ensure that the reactor coolant continues to meet these parameters, the chemical composition of the coolant will be analyzed periodically, as discussed in the Technical Requirements Manual.

The above requirements and procedures, coupled with the previously described design features for minimizing leakage, ensure that the valves will perform their intended functions.

5.4.12.4 Tests and Inspections

Technical Specification 3.4.14, RCS Pressure Isolation Valve (PIV) Leakage, requires leakage from reactor coolant pressure isolation valves to be within limits. The valves to which Technical specification 3.4.14 applies and which are required to satisfy leakage limits are stated in FSAR table 5.4.12-3.

FSAR table 5.4.12-3 shows the following:

- The valves to which Technical Specification 3.4.14 applies and the associated function.
- The size of each RCS pressure isolation valve, and
- The maximum allowable leakage for each RCS pressure isolation valve.

Hydrostatic shell test and seat leakage and functional tests are performed on all RCS valves. The tests and inspections discussed in section 3.10.N are performed to ensure the operability of the active valves.

There are no full-penetration welds within the valve body walls. Valves are accessible for disassembly and internal visual inspection to the extent practical. Plant layout configurations determine the degree of inspectability. The valve nondestructive examination program is given in table 5.4.12-2. Inservice inspection is discussed in subsection 5.2.4.

Tests and inspections performed under the following license renewal aging management programs are credited as applicable to various RCS valves:

- Bolting Integrity Program (see subsection 19.2.2).
- Boric Acid Corrosion Control Program (see subsection 19.2.3).
- External Surfaces Monitoring Program (see subsection 19.2.8).
- Inservice Inspection Program (see subsection 19.2.13).
- Oil Analysis Program (subsection 19.2.16).
- One-Time Inspection Program (see subsection 19.2.17).
- Water Chemistry Control Program (see subsection 19.2.28).

5.4.13 SAFETY AND RELIEF VALVES

5.4.13.1 Design Bases

The combined capacity of the pressurizer safety valves can accommodate the maximum pressurizer surge resulting from complete loss of load. Sizing of the pressurizer safety valves is discussed in subsection 5.2.2.

The pressurizer power-operated relief valves (PORVs) are designed to limit pressurizer pressure to a value below the high pressure reactor trip setpoint. They are designed to fail in the closed position on loss of actuating power.

5.4.13.2 Design Description

The pressurizer safety valves are of the pop type. The valves are spring loaded and self-actuated by direct fluid pressure and have backpressure compensation features.

The pipe connecting each pressurizer nozzle to its safety valve is shaped in the form of a loop seal. Condensate resulting from normal heat losses accumulates in the loop. This loop seal minimizes any leakage of hydrogen gas or steam through the safety valve seats. If the pressurizer pressure exceeds the set pressure of the safety valves, they start lifting, and the water from the seal discharges during the actuation period.

The pressurizer PORVs are solenoid-operated valves which respond to a signal from a pressure-sensing system or to manual control. Remotely operated block valves are provided to isolate the inlets to the PORVs if excessive leakage develops.

The PORVs and their associated block valves are interlocked by a pressurizer low-pressure interlock. Actuation of the interlock prevents the relief valves from opening and closes the block valves. Manual control may override this interlock.

In the event that a pressurizer PORV open signal actuation is sent due to a failure in a pressure channel associated with normal PORV operation, the interlock is provided to close the PORV as pressure decreases below the interlock pressure setpoint. The pressure signal associated with the interlock originates in the narrow range pressurizer pressure instrumentation. This signal and interlock operate separately from the cold overpressure pressure control signal, which originates in the wide range pressure instrumentation in the reactor coolant system loops. The overpressure protection system would not become disabled in the event of a single failure. The logic diagrams for the PORV interlocks are shown in drawing 1X6AA02-235.

In accordance with the requirements of NUREG-0737, TMI Action Item II.D.3, positive position indication is provided for the primary safety and relief valves.

Position indication on the PORV is accomplished through electrical reed switches. A magnetic rod, actuated by the valve plug, is located inside a projection above the top face of the bonnet and operates the reed switches contained in a switch assembly mounted externally on the bonnet. Safety valve indication is also accomplished through reed switches.

Temperatures in the pressurizer safety and relief valve discharge lines are measured, and an indication and a high alarm are provided on the main control board. An increase in a discharge line temperature is an indication of leakage or relief through the associated valve.

The PORVs provide the safety-related means for reactor coolant system depressurization to achieve cold shutdown. For a discussion of the use of these valves to achieve safety-grade cold shutdown, see subsection 5.4.7.

Design parameters for the pressurizer safety valves and power relief valves are given in table 5.4.13-1.

Relief and safety valve failures to close will promptly be reported to the Nuclear Regulatory Commission. This will meet the requirements of NUREG-0737, Action Item II.K.3.3.

5.4.13.3 <u>Design Evaluation</u>

The pressurizer safety valves prevent reactor coolant system pressure from exceeding 110% of system design pressure, in compliance with the ASME Code, Section III.

The slight time delay associated with the discharge of water from the loop seal piping configuration is accounted for in the limiting case analyses discussed in chapter 15.

The limiting pressure transient is the turbine trip event as described in subsection 15.2.3 and in the ASME Code Overpressure Protection Report. In the event, the pressurizer safety valves are assumed to open following a 2% tolerance in the set pressure, plus a 1% shift resulting from the presence of the loop seal, plus the time delay required to purge the loop seal. After these effects have been accounted for, the safety valves are assumed to relieve at their full capacity. As described in subsection 15.2.3, there is margin to the 110% of design pressure limit of 2748.5 psia. The results from the limiting case from subsection 15.2.3 are also presented in figure 5.4.13-1.

The pressurizer PORVs prevent actuation of the reactor high-pressure trip for all design transients up to and including the design step-load decreases with steam dump. The relief valves also limit undesirable opening of the spring-loaded safety valves.

5.4.13.4 Tests and Inspections

All safety and relief valves are subjected to hydrostatic tests, seat leakage tests, operational tests, and inspections, as required. For safety valves that are required to function during a faulted condition, additional tests are performed. These tests are described in section 3.10.N. There are no full-penetration welds within the valve body walls. Valves are accessible for disassembly and internal visual inspection. Refer to subsection 5.4.12 for the list of license renewal aging management programs credited to manage aging of various RCS valves.

Safety and relief valves similar to those at VEGP have been tested within the Electric Power Research Institute safety and relief test program and have been found adequate for steamflow and waterflow. The completion of this program addresses the requirements of Action Item II.D.1 of NUREG-0737 as related to valve testing.

5.4.14 COMPONENT SUPPORTS

5.4.14.1 Design Bases

Component supports allow unrestrained lateral thermal movement of the loop during plant operation and provide restraint to the loops and components during accident and seismic conditions. The loading combinations and design stress limits are discussed in paragraph 3.9.B.3.^(a) Support design is in accordance with the American Society of Mechanical Engineers

⁽a) Reference 1 provides the original criteria for postulating breaks in the reactor coolant loop. The basis for eliminating eight of these postulated large pipe breaks in the reactor coolant loop is provided in reference 2. The RCL component support design configuration of Unit 1 is unchanged from that provided in reference 1. However, the elimination of the large pipe breaks led to partial installation of the primary loop whip restraints on Unit 1 and the removal of the primary loop whip restraints on Unit 2 and the reduction from five large bore hydraulic snubbers to two in the steam generator upper support assemblies

(ASME) Code, Section III, Subsection NF. The design maintains the integrity of the RCS boundary for normal, seismic, and accident conditions and satisfies the requirements of the piping code. The results of piping and supports stress evaluation are presented in section 3.9.

Conformance with Regulatory Guides 1.124 and 1.130 is discussed in section 1.9.

5.4.14.2 Description

The support structures are welded structural steel sections. Linear-type structures (tension and compression struts, columns, and beams) are used in all cases except for the reactor vessel supports, which are plate-type structures. Attachments to the supported equipment are nonintegral type that are bolted to or bear against the components. The supports-to-concrete attachments are either anchor bolts or embedded fabricated assemblies.

The supports permit virtually unrestrained thermal growth of the supported systems but restrain vertical, lateral, and rotational movement resulting from seismic and pipe break loadings. This is accomplished using spherical bushings in the columns for vertical support and girders, bumper pedestals, hydraulic snubbers, and tie rods for lateral support.

Because of manufacturing and construction tolerances, ample adjustment in the support structures is provided to ensure proper erection alignment and fit-up. This is accomplished by shimming or grouting at the supports-to-concrete interface and by shimming at the supports-to-equipment interface.

The supports for the various components are described in the following paragraphs.

5.4.14.2.1 Reactor Pressure Vessel

Supports for the reactor vessel (figure 5.4.14-1) are individual, air-cooled, rectangular box structures beneath the vessel nozzles bolted to the primary shield wall concrete. Each box structure consists of a horizontal top plate that receives loads from the reactor vessel shoe, a horizontal bottom plate that transfers the loads to the primary shield wall concrete, and connecting vertical plates. The supports are air-cooled to maintain the supporting concrete temperature within acceptable levels.

5.4.14.2.2 Steam Generator

As shown in figure 5.4.14-2, the steam generator supports consist of the following elements:

A. Vertical Support

Four individual columns provide vertical support for each steam generator. These are bolted at the top to the steam generator and at the bottom to the concrete structure. Spherical ball bushings at the top and bottom of each column allow unrestrained lateral movement of the steam generator during heatup and cooldown. The column base design permits both horizontal and vertical adjustment of the steam generator for erection and adjustment of the system.

B. Lower Lateral Support

for Unit 2. The structural analysis of the RCL component supports for Units 1 and 2 considers the elimination of loads due to RCL breaks.

Lateral support is provided at the generator tube sheet by fabricated steel girders and struts. These are bolted to the compartment walls and include bumpers that bear against the steam generator but permit unrestrained movement of the steam generator during changes in system temperature. Stresses in the beams caused by wall displacements during compartment pressurization are considered in the design.

C. Upper Lateral Support

Upper lateral support of the steam generator is provided by a builtup ring plate girder at the operating deck. The 2-way acting snubbers restrain sudden seismic or blowdown-induced motion but permit the normal thermal movement of the steam generator.

Movement perpendicular to the thermal growth direction of the steam generator is prevented by struts.

5.4.14.2.3 Reactor Coolant Pump

Three individual columns, similar to those used for the steam generator, provide the vertical support for each pump. Lateral support for seismic and blowdown loading is provided by three lateral tension tie bars. The pump supports are shown in figure 5.4.14-3.

5.4.14.2.4 Pressurizer

The supports for the pressurizer, as shown in figure 5.4.14-4, consist of:

- A. A steel ring plate between the pressurizer skirt and the supporting structure. The ring serves as leveling and adjusting member for the pressurizer.
- B. The upper lateral support consists of struts cantilevered off the compartment walls that bear against the lugs provided on the pressurizer.

5.4.14.2.5 Control Rod Drive Mechanism (CRDM) Supports

The support system for the CRDM provides lateral restraint to limit CRDM deflections due to seismic or pipe break loadings. The CRDM support system consists of the following:

- A. A support platform extends across the top of the control rod drive housing columns. The CRDM rod travel housing extensions protrude through holes in this platform, thus limiting lateral deflection of the CRDM housing.
- B. Vertical support of the support platform is provided by three reactor vessel head lifting legs. The lifting legs are vertical columns which are pinned to the reactor vessel head.
- C. Horizontal support of the support platform is provided by lateral tension tie rods which are pinned to the refueling cavity wall.

5.4.14.3 <u>Evaluation</u>

Detailed evaluation ensures the design adequacy and structural integrity of the reactor coolant loop and the primary equipment supports system. This detailed evaluation is made by comparing the analytical results with established criteria for acceptability. Structural analyses are performed to demonstrate design adequacy for safety and reliability of the plant in case of a large or small seismic disturbance and/or loss-of-coolant accident conditions. Loads which the system is expected to encounter often during its lifetime (thermal, weight, and pressure) are applied, and stresses are compared to allowable values. The modeling and analysis methods are discussed in paragraph 3.9.N.1.4

5.4.14.4 <u>Tests and Inspections</u>

Nondestructive examinations are performed in accordance with the procedures of the ASME Code, Section V, except as modified by the ASME Code, Section III, Subsection NF. The Inservice Inspection Program is credited as a license renewal aging management program for RCS primary equipment supports (see subsection 19.2.13). The Structural Monitoring Program is also credited as a license renewal aging management program for ASME piping and component supports (see subsection 19.2.32).

5.4.14.5 References

- 1. "Pipe Breaks for the LOCA Analysis of the Westinghouse Primary Coolant Loop," WCAP-8082-P-A (proprietary) and WCAP-8172-A (nonproprietary), January 1975.
- 2. Federal Register, Vo. 50, No. 27, February 8, 1985.

5.4.14.6 Bibliography

DeRosa, P., et al., "Evaluation of Steam Generator Tube, Tube Sheet and Divider Plant Under Combined LOCA Plus SSE Conditions," WCAP-7832, December 1973.

Eggleston, F. T., "Safety-Related Research and Development for Westinghouse Pressurized Water Reactor, Program Summaries-Winter 1976," <u>WCAP-8768</u>, Revision 1, June 1977.

"Reactor Coolant Pump Integrity in LOCA," WCAP-8163, September 1973.

5.4.15 REACTOR VESSEL HEAD VENT SYSTEM

The reactor vessel head vent system (RVHVS) (figure 5.1.2-1) removes noncondensable gases or steam from the reactor vessel head. This system is designed to mitigate a possible condition of inadequate core cooling or impaired natural circulation resulting from the accumulation of noncondensable gases in the reactor coolant system (RCS). The design of the RVHVS is in accordance with the requirements of TMI action plan item II.B.1 of NUREG 0737 as discussed below.

5.4.15.1 <u>Design Bases</u>

The RVHVS is designed to remove noncondensable gases or steam from the RCS via remote manual operations from the control room. The system discharges to the pressurizer relief tank. Additionally, a letdown flow path is provided from the reactor vessel head vent to the excess letdown heat exchanger in the chemical and volume control system. The RVHVS is designed to vent a volume of hydrogen at system design pressure and temperature approximately equivalent to one-half of the RCS volume in 1 h.

The system provides for venting the reactor vessel head by using only safety grade equipment. The RVHVS satisfies applicable requirements and industry standards, including ASME Code classification, safety classification, single-failure criteria, and environmental qualification.

All piping and equipment from the vessel head vent up to and including the second isolation valve in each flow path are designed and fabricated in accordance with ASME Section III, Class 1 requirements. The piping and equipment in the flow paths from the isolation valve to the modulating valves and from the isolation valves to the excess letdown heat exchanger are designed and fabricated in accordance with ASME Section III, Class 2 requirements. The remainder of the piping and equipment is Seismic Category 1, nonnuclear safety.

All supports and support structures conform with the requirements of the ASME Code.

The analysis of the reactor vessel head vent piping is based on the following plant operating conditions defined in the ASME Code, Section III:

A. Normal Condition

Pressure, deadweight, and thermal expansion analysis of the vent piping during:

- 1. Normal reactor operation with the vent isolation valves closed.
- 2. Post-refueling venting.
- B. Upset Condition

Loads generated by the operating basis earthquake (OBE).

C. Faulted Condition

Loads generated by the safe shutdown earthquake (SSE). Loads generated by valve thrust during venting. In accordance with ASME III, faulted conditions are not included in fatigue evaluations.

The Class 1 piping used for the reactor vessel head vent is 1 in. schedule 160 and, therefore, in accordance with ASME III, is analyzed following the procedures of NC-3600 for Class 2 piping.

For all plant operating conditions listed above, the piping stresses are shown to meet the requirements of equations 8, 9, and 10 or 11 of ASME III, NC-3600, with a design temperature of 650°F and a design pressure of 2,485 psig.

5.4.15.2 <u>System Description</u>

The RVHVS consists of a single active failure proof flow path with redundant isolation valves. The equipment design parameters are listed in table 5.4.15-1.

The active portion of the system consists of four 1-in. open/close solenoid-operated isolation valves connected to the existing 1-in. vent pipe, which is located near the center of the reactor vessel head. The system design with two valves in series in each flow path minimizes the possibility of reactor coolant pressure boundary leakage. The isolation valves in one flow path

are powered by one vital power supply and the valves in the second flow path are powered by a second vital power supply. The isolation valves are fail closed normally closed valves. The valves are included in the valve operability program and will be qualified to IEEE-323-1975, - 344-A75 and 382-1972. The control valves are also normally closed, fail closed.

The vent system piping is supported to ensure that the resulting loads and stresses on the piping and on the vent connection to vessel head are acceptable.

5.4.15.3 <u>Safety Evaluation</u>

If one single active failure prevents a venting operation through one flow path, the redundant path is available for venting. The two isolation valves in each flow path provide a similar method of isolating the venting system. With two valves in series, the failure of any one valve or power supply will not inadvertently open a vent path or prevent opening and closing a flow path. Thus, the combination of safety grade train assignments and valve failure modes will not prevent vessel head venting nor venting isolation with any single active failure.

The RVHVS has two normally deenergized valves in series in each flow path. This arrangement eliminates the possibility of an opened flow path due to the spurious movement of one valve. As such, power lockout to any valve is not considered necessary.

A break of the RVHVS line would result in a small loss-of-coolant accident (LOCA) of not greater than 1-in. diameter. Such a break is similar to those analyzed in WCAP-9600 (1979). Since a break in the head vent line would behave similarly to the hot leg break case presented in WCAP-9600, the results presented therein are applicable to a RVHVS line break. Therefore, this postulated vent line break results in no calculated core uncovery.

5.4.15.4 Inspection and Testing Requirements

Inservice inspection is conducted in accordance with section 6.6.

5.4.15.5 Instrumentation Requirements

The system is operated from the control room or the shutdown panels. The isolation valves have stem position switches. The position indication from each valve is monitored at the control room by status lights.

TABLE 5.4.1-1 (SHEET 1 OF 2)

REACTOR COOLANT PUMP DESIGN PARAMETERS

Unit design pressure (psig) Unit design temperature (°F) Unit overall height (ft)	2485 650 ^(a) 27.4
Seal water injection (gal/min)	8
Seal water return (gal/min)	3
Component cooling waterflow (gal/min)	596
Maximum continuous component cooling	
water inlet temperature (°F)	105
Total weight, dry (lb)	201,300

<u>Pump</u>

Design flow (gal/min)	100,600
Developed head (ft)	288
NPSH required (ft)	Figure 5.4.1-2
Suction temperature, thermal design (°F)	558.2
Pump discharge nozzle, inside diameter (in.)	27-1/2
Pump suction nozzle, inside diameter (in.)	31
Speed (rpm)	1187
Water volume (ft³)	80 ^(b)

Motor

Туре	Drip-proof squirrel cage induction, with water/air coolers
Power (hp)	7000
Voltage (V)	13,200
Phase	3
Frequency (Hz)	60
Insulation class	Class F
	thermalastic
	epoxy
	insulation

Current (amp)
Starting
Nominal input, hot reactor coolant
Nominal input, cold reactor coolant
336

TABLE 5.4.1-1 (SHEET 2 OF 2)

Pump moment of inertia, maximum (lb/ft²)

Flywheel	·	,	70,000
Mótor			22,500
Shaft			520
Impeller			1980

a. Design temperature of pressure-retaining parts of the pump assembly exposed to the reactor coolant and injection water on the high-pressure side of the controlled leakage seal is that temperature determined for the parts for a reactor coolant loop temperature of 650°F.

b. Composed of reactor coolant in the casing and of seal injection and cooling water in the thermal barrier.

TABLE 5.4.1-2 REACTOR COOLANT PUMP QUALITY ASSURANCE PROGRAM

	RT ^(a)	<u>UT</u> ^(a)	PT ^(a)	MT ^(a)
Castings	Yes		Yes	
Forgings Main shaft Main studs		Yes Yes	Yes	Yes
Plate Flywheel		Yes	Yes ^(b)	Yes ^(b)
Weldments Circumferential Instrument connections	Yes		Yes Yes	

a.

RT - Radiographic. UT - Ultrasonic. PT - Dye penetrant. MT - Magnetic particle.

Of machined bores keyways and drilled holes (either PT or MT). b.

TABLE 5.4.2-1 STEAM GENERATOR DESIGN DATA

Design pressure, reactor coolant side (psig)	2485
Design pressure, steam side (psig)	1185
Design pressure, primary to secondary (psi)	1600
Design temperature, reactor coolant side (°F)	650
Design temperature, steam side (°F)	600
Design temperature, primary to secondary (°F)	650
Total heat transfer surface area (ft²)	55,000
Maximum moisture carryover (weight percent)	0.25
Overall height (ft-in.)	67-8
Number of U-tubes	5626
U-tube nominal diameter (in.)	0.688
Tube-wall nominal thickness (in.)	0.040
Number of manways	4
Inside diameter of manways (in.)	16
Number of handholes	8
Design fouling factor (ft²-h-°F/Btu)	0.00006
Steamflow (lb/h)	4.055 x 10 ⁶ to 4.080 x 10 ⁶

TABLE 5.4.2-2 (SHEET 1 OF 2)

STEAM GENERATOR QUALITY ASSURANCE PROGRAM

	RT ^(a)	<u>UT</u> ^(a)	PT ^(a)	MT ^(a)	ET ^(a)
Tube Sheet Forging Cladding		Yes Yes ^(b)	Yes	Yes	
<u>Channel Head</u> (if fabricated) Fabrication Cladding	Yes ^(c)	Yes ^(d)	Yes	Yes	
Secondary Shell and Head Plates		Yes			
<u>Tubes</u>		Yes			Yes
Nozzles (Forgings)		Yes		Yes	
Weldments Shell, longitudinal Shell, circumferential Cladding (channel head-tube sheet joint cladding restoration)	Yes Yes		Yes	Yes Yes	
Primary nozzles to	Yes			Yes	
fab head Manways to fab head	Yes			Yes	
Steam and feedwater nozzles to shell	Yes			Yes	
Support brackets			Yes		
Tube to tube sheet			Yes		
Instrument connections (primary and secondary)				Yes	
Temporary attachments after removal				Yes	

TABLE 5.4.2-2 (SHEET 2 OF 2)

	RT ^(a)	<u>UT</u> ^(a)	PT ^(a)	MT ^(a)	ET ^(a)
After hydrostatic test (all major pressure boundary welds and complete cast channel head - where accessible)				Yes	Yes
Nozzle safe ends (if weld deposit)	Yes		Yes		

- Weld deposit. C.
- Base material only. d.

RT - Radiographic. UT - Ultrasonic. PT - Dye penetrant. MT - Magnetic particle. ET - Eddy current.

b. Flat surfaces only.

TABLE 5.4.3-1 REACTOR COOLANT PIPING DESIGN PARAMETERS

Reactor Inlet Piping Inside diameter (ID) (in.)	27 1/2
Reactor Inlet Piping Nominal wall thickness (in.)	2.32
Reactor Outlet Piping ID (in.)	29
Reactor Outlet Piping Nominal wall thickness (in.)	2.45
Coolant Pump Suction Piping ID (in.)	31
Coolant Pump Suction Piping Nominal wall thickness (in.)	2.60
Pressurizer Surge Line Piping Nominal pipe size (in.)	16 reduced to 14
Pressurizer Surge Line Piping Nominal wall thickness, for 16 in. nominal pipe size, schedule 160 (in.) Nominal wall thickness, for 14 in. nominal pipe size, schedule 160 (in.)	1.594 1.406
Nominal Water Volume, all four loops including surge line (ft ³)	1200-1300
RCL Piping Design/operating pressure (psig) Design temperature (°F)	2485/2235 650
Pressurizer Surge Line Design pressure (psig) Design temperature (°F)	2485 680
Pressurizer Safety Valve Inlet Line Design pressure (psig) Design temperature (°F)	2485 680
Pressurizer Power-Operated Relief Valve Inlet Line Design pressure (psig) Design temperature (°F)	2485 680

TABLE 5.4.3-2 REACTOR COOLANT PIPING QUALITY ASSURANCE PROGRAM

	RT ^(a)	<u>UT</u> ^(a)	PT ^(a)
Fittings and Pipe (Castings)	Yes		Yes
Fittings and Pipe (Forgings)		Yes	Yes
Weldments			
Circumferential	Yes		Yes
Nozzle to runpipe (except no RT for nozzles less than 6 in.)	Yes		Yes
Instrument connections			Yes
Castings	Yes		Yes (after finishing)
Forgings			Yes (after finishing)

a. RT - Radiographic; UT - ultrasonic; PT - dye penetrant.

TABLE 5.4.7-1

DESIGN BASES FOR RHRS OPERATION

~4	
~365	
~350	
105 ^(a)	
~16 ^(b)	
140 ^(b)	
80.4 x 10 ⁶	
2.5 x 10 ⁶ Btu/h°F	
0.0003 ft ² h°F/Btu	
	~365 ~350 105 ^(a) ~16 ^(b) 140 ^(b) 80.4 x 10 ⁶ 2.5 x 10 ⁶ Btu/h°F

a. The maximum CCW temperature during cooldown with offsite power available is 120°F.

b. The design bases for cold shutdown are 32 h (one train) to 200°F.

TABLE 5.4.7-2

RHRS COMPONENT DATA

2

RHR Pumps

Number

Number	2
Design pressure (psig)	600
Design temperature (°F)	400
Design flow (gal/min)	3000
Design head (ft)	375
Material	Austenitic
	stainless
	steel

Residual Heat Exchangers

Design heat removal capacity (Btu/h) Estimated UAF _{LMTD} (Btu/h)		32.8 x 10 ⁶ 2.5 x 10 ⁶
	Tube Side	Shell Side
Design pressure (psig) Design temperature (°F) Design flow (lb/h) Inlet temperature (°F) Outlet temperature (°F)	600 400 1.48 x 10 ⁶ 140 117.8	150 200 2.48 x 10 ⁶ 105 118.2
Material	Austenitic stainless steel	Carbon steel
Fluid	Reactor coolant	CCW

TABLE 5.4.7-3 (SHEET 1 OF 5)

FAILURE MODES AND EFFECTS ANALYSIS - RESIDUAL HEAT REMOVAL SYSTEM ACTIVE COMPONENTS - NORMAL COOLDOWN OPERATION

Remarks	Valve is electrically interlocked with the containment sump suction valve HV-8811A, with RWST isolation valve HV-8812A, with RHR to charging pump suction isolation valve HV-8804A and with a "prevent-open" pressure interlock PT-438 (PT-408). The valve cannot be opened remotely from the CB if one of the indicated isolation valves is open or if RC loop pressure exceeds 365 psig.	If both trains of RHRS are unavailable for plant cooldown due to multiple component failures, the auxiliary feedwater system and SG power-operated relief valves can be used to perform the safety function of removing residual heat.	Same remarks as those stated for item 1, except for pressure interlock PT-418 (PT-428) control.
Failure Detection Method ^(b)	Valve position indication (closed to open position change) at CB; RC loop 1 or 4 hot leg pressure indication at CB; RHR train A discharge flow indication; and RHR pump discharge pressure indication at CB.		Same methods of detection as those stated for item 1.
Effect on System Operation ^(a)	Failure blocks reactor coolant flow from hot leg of RC loop 1 through train A of RHRS. Fault reduces redundancy of RHR coolant trains provided. No effect on safety for system operation. Plant cooldown requirements will be met by reactor coolant flow from hot leg of RC loop 4 flowing through train B of RHRS. However, time required to reduce RCS temperature will be extended.		Same effect on system operation as that stated for item 1.
Failure Mode	a. Fails to open on demand (open manual mode CB switch selection).		Same failure modes as those stated for item 1.
Component	1. Motor-operated gate valve HV-8701A (HV-8701B analogous).		2. Motor-operated gate valve HV-8702A (HV-8702B analogous).

TABLE 5.4.7-3 (SHEET 2 OF 5)

Remarks	The RHRS shares components with the ECCS. Pumps are tested as part of the ECCS testing program. (See subsection 6.3.4.) Pump failure may also be detected during ECCS testing.	Valve is automatically controlled to open when pump discharge is less than the open setpoint (824 gpm at 350°F, 780 gpm at 100°F) and close when the discharge exceeds the closed setpoint (1944 gpm at 350°F, 1841 gpm at 100°F). The valve protects the pump from deadheading during ECCS operation CB switch set	to "Auto" position for automatic control of valve positioning.
Failure Detection Method	Open pump switchgear circuit breaker indication at CB; circuit breaker close position monitor light for group monitoring of components at CB; common breaker trip alarm at CB; RC loop 1 hot leg pressure indication at CB; RHR train A discharge flow indication and low flow alarm at CB; and pump discharge pressure indication at CB; MHR train A discharge flow indication and low flow alarm at CB; and pump discharge pressure indication at CB.	Valve position indication (closed to open position change) at CB.	Valve position indication (open to closed position change) and RHRS train A discharge flow indication at CB.
Effect on System Operation	Failure results in loss of reactor coolant flow from hot leg of RC loop 1 through train A of RHRS. Fault reduces redundancy of RHR coolant trains provided. No effect on safety for system operation. Plant cooldown requirements are met by reactor coolant flow from hot leg or RC loop 4 flowing through train B of RHRS. However, time required to reduce RCS temperature will be extended.	Failure blocks miniflow line to suction of RHR pump 1 during cooldown operation. No effect on safety for system operation. Operator may establish miniflow for RHR pump 1 operation by opening of CVCS letdown control valve HCV-128 and manual valve 1205-U4-021 to allow flow to CVCS.	Failure allows for a portion of RHR heat exchanger discharge flow to be bypassed to suction of RHR pump 1. RHRS train A is degraded for the regulation of coolant temperature by RHR heat exchanger 1. No effect on safety for system operation. Cooldown of RCS within established specification cooldown rate may be accomplished through operator action of throttling flow control valve HCV-606 and controlling cooldown with redundant RHRS train B.
Failure Mode	Fails to deliver working fluid.	a. Fails to open on demand (open manual mode CB switch selection).	b. Fails to close on demand (auto mode CB switch selection).
Component	3. RHR pump 1, (RHR pump 2 analogous).	4. Motor-operated globe valve FV-610 (FV-611 analogous).	

TABLE 5.4.7-3 (SHEET 3 OF 5)

Remarks	Valve is designed to fail closed and is electrically wired so that electrical solenoid of the air diaphragm operator is energized to open the valve. Valve is normally closed to align RHRS for ECCS operation during plant power operation and load follow.	
Failure Detection Method	RHR pump 1 discharge flow temperature and RHRS train A discharge to RCS cold leg flow temperature recorded on the plant computer; and RHRS train A discharge to RCS cold leg flow indication at CB.	Same methods of detection as those stated for item 5.a.
Effect on System Operation	Failure prevents coolant discharged from RHR pump 1 from bypassing RHR heat exchanger 1 resulting in mixed mean temperature of coolant flow to RCS being low. RHRS train A is degraded for the regulation of controlling temperature of coolant. No effect on safety for system operation. Cooldown of RCS within established specification rate may be accomplished through operator action of throttling flow control valve HCV-606 and controlling cooldown with redundant RHRS train B.	Failure allows coolant discharge from RHR pump 1 to bypass RHR heat exchanger 1, resulting in mixed mean temperature of coolant flow to RCS being high. RHRS train A is degraded for the regulation of controlling temperature of controlling temperature of coolant. No effect on safety for system operation. Cooldown of RCS within established specification rate may be accomplished through operator action of throttling flow control valve HCV-606 and controlling cooldown with redundant RHRS train B. However, cooldown time will be extended.
Failure Mode	a. Fails to open on demand (Auto mode CB switch selection).	b. Fails to close on demand (Auto mode CB switch selection).
Component	5. Air diaphragm- operated butterfly valve FCV-618 (FCV-619 analogous).	

TABLE 5.4.7-3 (SHEET 4 OF 5)

Remarks	Valve is designed to fail open. Valve is normally open to align RHRS for ECCS operation during plant power operation and load follow.		Valve is a component of the ECCS that performs an RHR function during plant cooldown. Valve is normally open to align RHRS for ECCS operation during plant power operation and load follow.
Failure Detection Method	Same methods of detection as those stated for item 5. In addition, monitor light and alarm (valve closed) for group monitoring of components at CB.	Same methods of detection as those stated for item 6.a.	Valve position indication (open to closed position change) at CB and valve (closed) monitor light and alarm at CB.
Effect on System Operation	Failure prevents control of coolant discharge flow from RHR heat exchanger 1, resulting in loss of mixed mean temperature coolant flow adjustment to RCS. No effect on safety for system operation. Cooldown of RCS within established specification rate may be accomplished by operator action of controlling cooldown with redundant RHRS train B.	Same effect on system operation as that stated for item 6.a.	Failure reduces the redundancy of isolation valves provided to isolate RHRS train A from RWST. No effect on safety for system operation. Check valve in series with motor-operated valve provides the primary isolation against the bypass of RCS coolant flow from the suction of RHR pump A to RWST.
Failure Mode	a. Fails to close on demand for flow reduction.	b. Fails to open on demand for increased flow.	Fails to close on demand.
Component	6. Air diaphragm- operated butterfly valve HCV-606 (HCV-607 analogous).		7. Motor-operated gate valve HV-8812A (HV-8812B analogous).

TABLE 5.4.7-3 (SHEET 5 OF 5)

Component	Failure Mode	Effect on System Operation	Failure Detection Method	Remarks
8. Motor-operated gate valve HV-8716A (HV-8716B analogous).	Fails to close on demand.	Failure reduces the redundancy of isolating, train A from train B for single train operation of RHRS. No effect on safety for system operation. Isolation will be provided by closing valve HV-8716B (HV-8716A).	Same as item 7.	

Automatic.
Main control board.
Chemical and volume control system.
Emergency core cooling system.
Reactor coolant.
Reactor coolant system.
Residual heat removal.
Residual heat removal.
Residual heat removal.
Refueling water storage tank.
Steam generator.

b. As part of plant operation; periodic tests, surveillance inspections, and instrument calibrations are conducted to monitor equipment and performance. Failures may be detected during such monitoring of equipment, in addition to detection methods noted.

a. List of acronyms and abbreviations.

TABLE 5.4.7-4 (SHEET 1 OF 5)

RESIDUAL HEAT REMOVAL SYSTEM - SAFETY GRADE COLD SHUTDOWN OPERATIONS - FAILURE MODES AND EFFECTS ANALYSIS

Remarks	Valve is electrically interlocked with a containment suction valve HV-8811A RWST to RHR suction line isolation valve HV-8812A, with RHR to charging pump suction line isolation valve HV-8804A and with a "prevent-open" pressure interlock PT-438 (PT-408). The valve can not be opened remotely from the CB if one of the indicated isolation valves is open or if RC loop pressure exceeds 365 psig. The valve can be manually opened.	
Failure Detection Methods ^(c)	a. Valve open/close position indication at CB; RC loop 1 or 4 hot leg pressure indication at CB; RHR train A discharge flow indication, and RHR pump 1 discharge pressure indication at CB.	b. Valve is interlocked with pressure interlock PT-438 (PT-408) to alarm on the main control board annunciator panel if one or both of the valves is not fully closed and RCS pressure exceeds 420 psig.
Effect on System Operation	a. Failure blocks reactor coolant flow from hot leg of RC loop 1 through train A of RHRS. Failure reduces redundancy of RHR coolant trains provided. No effect on safety for system operation. Plant cooldown requirements are met by reactor coolant flow from hot leg of RC loop 4 flowing through train B of RHRS, however time required to reduce RCS temperature is extended.	b. Failure reduces redundancy of main control board annunciator alarm at 420 psig. No effect on safety system operation. Plant operating procedures require that operators close both valves prior to an RCS pressure of 420 psig. Alternate alarm is provided by valve HV-8701B (HV-8701A).
Function ^(b)	Provides isolation of fluid flow from the RCS to the suction of RHR pump 1.	
Failure Mode	a. Fails to open on demand.	b. Once the valves are open the main control board annunciator alarm fails and RCS pressure exceeds 420 psig.
<u>Component^(a)</u>	1. Motor-operated gate valve HV-8701A (HV-8701B analogous).	

TABLE 5.4.7-4 (SHEET 2 OF 5)

Remarks	Same as item 1, except for pressure interlock PT418 (PT428) control.	The RHRS shares components with the ECCS. Pumps are tested as part of the ECCS testing program. (See subsection 6.3.4.)	Valve is automatically controlled to open when pump discharge is less than the open setpoint (824 gpm at 100°F) and close when the discharge exceeds the closed setpoint (1944 gpm at 350°F, 1841 gpm at 100°F).
Failure Detection Methods	Same as item 1.	Open pump switchgear circuit breaker indication at CB; circuit breaker close position monitor light for group monitoring of components at CB; common breaker trip alarm at CB; RC loop 1 hot leg pressure indication at CB; RHR train A discharge flow indication and low flow alarm at CB; and pump discharge pressure indication at CB; and pump discharge pressure indication at CB.	a. Valve open/close position indication at CB; and RHRS train A discharge flow indication at CB.
Effect on System Operation	Same as item 1.	Failure results in loss of reactor coolant flow from hot leg of RC loop 1 through train A of RHRS. Failure reduces redundancy of RHR coolant trains provided. No effect on safety for system operation. Plant cooldown requirements are met by reactor coolant flow from hot leg of RC loop 4 flowing through train B of RHRS, however, time required to reduce RCS temperature is extended.	a. Failure blocks miniflow line to suction of RHR pump 1 during cooldown operation. No effect on safety for system operation. Plant cooldown requirements are met by reactor coolant flow from hot leg of RC loop 4 flowing through train B of RHRS. However, time required to reduce RCS temperature is extended.
Function	Same as item 1.	Provides fluid flow of reactor coolant through RHR heat exchanger 1 to reduce RCS temperature during cooldown operation.	Provides regulation of fluid flow through miniflow bypass line to suction of RHR pump 1 to protect against overheating of the pump and loss of discharge flow from the pump.
Failure Mode	Same as item 1.	Fails to deliver working fluid.	a. Fails closed.
Component	2. Motor-operated gate valve HV-8702A (HV-8702B analogous).	3. RHR pump 1 (RHR pump 2 analogous).	4. Motor-operated globe valve FV-610 (FV-611 analogous).

TABLE 5.4.7-4 (SHEET 3 OF 5)

Remarks		Valve is designed to fail closed and is electrically wired so that electrical solenoid of the air diaphragm operator is energized to open the valve. Valve is normally closed to align RHRS for ECCS operation during plant power operation and load follow. Valve is designed for normal plant cooldown operation. It is not required for safety grade cold shutdown operation.
Failure Detection Methods	Same as item 4.a.	a. RHR pump 1 discharge flow temperature and RHRS train A discharge to RCS cold leg flow temperature recorded on the plant computer; and RHRS train A discharge to RCS cold leg flow indication at CB.
Effect on System Operation	b. Failure allows for a portion of RHR heat exchanger 1 discharge flow to be bypassed to suction of RHR pump 1. RHRS train A is degraded for the regulation of coolant temperature by RHR heat exchanger 1. No effect on safety for system operation. Cooldown of RCS remains within established specification cooldown rate.	ant discharged from RHR pump 1 from bypassing RHR heat exchanger 1 resulting in mixed mean temperature of coolant flow to RCS being low. RHRS train A is degraded for the regulation of controlling temperature of coolant. No effect on safety for system operation. Cooldown of RCS within established specification rate may be accomplished through operator action of throttling flow control valve HCV-606 and controlling cooldown with redundant RHRS train B.
Function		Controls rate of fluid flow bypassed around RHR heat exchanger 1 during cooldown operation.
Failure Mode	b. Fails open.	a. Fails to open on demand for flow increase ("Auto" mode CB switch selection).
Component		5. Air - diaphragm- operated - butterfly valve FCV-618 (FCV-619 analogous).

TABLE 5.4.7-4 (SHEET 4 OF 5)

Remarks		Valve is designed to fail open. Valve is normally open to align RHRS for ECCS operation during plant power operation and load follow.	
Failure Detection Methods	b. Same as item 5.a.	a. Same methods of detections as those stated for item 5.a. addition, monitor light and alarm (valve closed) for group monitoring of components at CB.	b. Same as item 6.a.
Effect on System Operation	discharged from RHR pump 1 to bypass RHR heat exchanger 1 resulting in mixed mean temperature of coolant flow to RCS being high. RHRS train A is degraded for the regulation controlling temperature of coolant. No effect on safety for system operation. Cooldown of RCS within established specification rate may be accomplished through operator action of throttling flow control valve HCV-606 and controlling cooldown with redundant RHRS train B; however, cooldown time is extended.	a. Failure prevents control of coolant discharge flow from RHR heat exchanger 1 resulting in loss of mixed mean temperature coolant flow adjustment to RCS. No effect on safety for system operation. Cooldown of RCS within established specification rate may be accomplished by operator action of controlling cooldown with redundant RHS train B.	b. Same as item 6.a.
Function		Controls rate of fluid flow through RHR heat exchanger 1 during cooldown operation.	
Failure Mode	b. Fails to close on demand for flow reduction (Auto mode CB switch selection).	a. Fails to close on demand for flow reduction.	b. Fails to open on demand for flow increase.
Component		6. Air diaphragm- operated butterfly valve HCV-606 (HCV-607 analogous).	

TABLE 5.4.7-4 (SHEET 5 OF 5)

Remarks	Valve is normally open to align RHRS for ECCS operation during plant power operation and load follow. Valve must be closed during plant cooldown to satisfy electrical interlock to permit valves HV-8701A and B (HV-8701A B) to be opened.	
Failure Detection Methods	Valve open/closed position indication at CB and valve (closed) monitor light and alarm at CB.	Same as item 7.
Effect on System Operation	No effect on safety for system operation. Plant cooldown requirements are met by reactor coolant flow from hot leg loop 4 flowing through train B of RHRS; however, time required to reduce RCS temperature is extended.	Failure reduces the redundancy for isolating RHR trains during cooldown. Negligible effect on system operation. Isolation valve HV-8716B (HV-8716A) provides backup isolation between the two RHR trains.
Function	Provides isolation of fluid from the RWST to suction of RHR pump 1 during cooldown operation.	Provides separation between the two RHR trains during cooldown operation.
Failure Mode	Fails to close on demand.	Fails to close on demand.
Component	7. Motor-operated gate valve HV-8812A (HV-8812B analogous).	8. Motor-operated gate valve HV-8716A (HV-8716B analogous).

List of acronyms and abbreviations. <u>.</u>

Anto		Automatic.	HELB		High-energy line break.	RWST	Refueling water storage tank.
BAT	•	Boric acid tank.	MELB		Moderate-energy line break.	Z S	Reactor vessel.
BIT	ı	Boron injection tank (Unit 1 only).	PRT		Pressurizer relief tank.	S	Safety injection.
CB	•	Main control board.	RC		Reactor coolant.	VCT	Volume control tank.
CVCS	•	Chemical and volume control system.	RCS	,	Reactor coolant system.		
ECCS	ì	Emergency core cooling system.	RHR		Residual heat removal.		

As part of plant operation, periodic tests, surveillance inspections, and instrument calibrations are made to monitor equipment and performance. Failures may be detected during such monitoring of equipment in addition to detection methods noted. ပ

Component 7 is a component of the ECCS that performs a safety-grade cold shutdown function.

TABLE 5.4.10-1

PRESSURIZER DESIGN DATA

Design pressure (psig)	2485
Design temperature (°F)	680
Surge line nozzle diameter (in.)	14
Heatup rate of pressurizer using heaters	
only (°F/h)	55
Internal volume (ft ³)	1800
Nominal conditions at 100-percent rated load	
Steam volume (ft ³)	720
Water volume (ft³)	1080

TABLE 5.4.10-2

REACTOR COOLANT SYSTEM DESIGN PRESSURE SETTINGS

	<u>Psig</u>
Hydrostatic test pressure	3106
Design pressure Safety valves (begin to open) High pressure reactor trip High pressure alarm Power-operated relief valves PV-0455A PV-0456A	2485 2460 2385 2310 2345 ^(a) 2335 ^(b)
Pressurizer spray valves (full open) Pressurizer spray valves (begin to open) Proportional heaters (begin to operate) Operating pressure Proportional heater (full operation) Backup heaters on Low pressure alarm Pressurizer power-operated relief valve interlock	2310 2260 2250 2235 2220 2210 2210 2185
Low pressure reactor trip	1870

a. At 2345 psi, a pressure signal initiates actuation (opening) of this valve. Remote manual control is also provided.

b. At 2335 psi, a pressure signal initiates actuation (opening) of this valve. Remote manual control is also provided.

TABLE 5.4.10-3

PRESSURIZER QUALITY ASSURANCE PROGRAM

	RT ^(a)	<u>UT</u> ^(a)	PT ^(a)	MT ^(a)
Heads				
Plates Cladding		Yes	Yes	
Shell				
Plates Cladding		Yes	Yes	
Heaters				
Tubing Centering of element	Yes	Yes ^(b)	Yes	
Nozzle (Forgings)		Yes	Yes ^(c)	Yes ^(c)
Weldments				
Shell,longitudinal Shell, circumferential Cladding	Yes Yes		Yes	Yes Yes
Nozzle safe end Instrument connection	Yes		Yes Yes	
Support skirt, longitudinal seam	Yes			Yes
Support skirt to lower head		Yes		Yes
Temporary attachments (after removal)				Yes
All external pressure boundary welds after shop hydrostatic tests				Yes

a. RT - Radiographic.

UT - Ultrasonic.

PT - Dye Penetrant.

MT - Magnetic Particle.

b. Eddy current testing can be used in lieu of UT.

c. MT or PT.

TABLE 5.4.11-1

PRESSURIZER RELIEF TANK DESIGN DATA

Design pressure (psig)	100
Normal operating pressure (psig)	3
Final operating pressure (psig) ^(a)	50
Rupture disc release pressure (psig)	
Nominal	91
Range	86 to 100
Normal water volume (ft³)	1350
Normal gas volume (ft³)	450
Design temperature (°F)	340
Initial operating water temperature (°F) (a)	120
Final operating water temperature (°F) (a)	200
Total rupture disc relief	1.6 x 10 ⁶
capacity at 100 psig (lb/h)	
Cooling time required following maximum	
discharge, approximately (h)	
Spray feed and bleed	1
Utilizing RCDT heat exchanger	8

a. For the design basis pressurizer steam discharge.

TABLE 5.4.11-2

DISCHARGES TO THE PRESSURIZER RELIEF TANK

Reactor coolant system (drawings 1X4DB111, 2X4DB111, 1X4DB112, 2X4DB112, and 1X4DB113)

Two pressurizer PORVs
Three pressurizer safety valves
One reactor vessel head vent

Residual heat removal system (drawing 1X4DB122)

Two suction line relief valves from the RCS hot legs

Chemical and volume control system (drawing 1X4DB114)

One seal water return line relief valve
One RCS letdown line relief valve

TABLE 5.4.11-3

PRESSURIZER RELIEF DISCHARGE SYSTEM NONDESTRUCTIVE TESTING PROGRAM

<u>Components</u>	Radiographic	<u>Ultrasonic</u>	Dye <u>Penetrant</u>
Fittings and pipe (castings)	Yes		Yes
Fittings and pipe (forgings)		Yes	Yes
Weldments Circumferential Nozzle to runpipe (except no radiographic for nozzles less than 4 in.)	Yes Yes		Yes Yes
Instrument connections			Yes

TABLE 5.4.12-1

REACTOR COOLANT SYSTEM VALVE DESIGN PARAMETERS

Design pressure (psig)	2485
Preoperational plant hydrotest (psig)	3106
Design temperature (°F)	650

TABLE 5.4.12-2

REACTOR COOLANT SYSTEM VALVES NONDESTRUCTIVE **EXAMINATION PROGRAM**

	RT ^(a)	<u>UT</u> ^(a)	<u>PT</u> ^(a)
Castings Larger than 4 in. 2 to 4 in.	Yes Yes ^(b)		Yes Yes
Forgings Larger than 4 in. 2 to 4 in.	(c)	(c)	Yes Yes

RT - Radiographic UT - Ultrasonic PT - Dye Penetrant a.

Weld ends only. b.

Either RT or UT. C.

TABLE 5.4.12-3 (SHEET 1 OF 2)

REACTOR COOLANT SYSTEM PRESSURE ISOLATION VALVES

	VALVE <u>NUMBER</u>	VALVE SIZE (in.)	<u>FUNCTION</u>	MAXIMUM ALLOWABLE <u>LEAKAGE(gpm)</u>
1.	HV-8701A	12	RHR Suction (gate valve)	5.0
2.	HV-8701B	12	RHR Suction (gate valve)	5.0
3.	HV-8702A	12	RHR Suction (gate valve)	5.0
4.	HV-8702B	12	RHR Suction (gate valve)	5.0
5.	1204-U4-120	2	SI-Hot Leg 2nd Isolation Valve	1.0
6.	1204-U4-121	2	SI-Hot Leg 2nd Isolation Valve	1.0
7.	1204-U4-122	2	SI-Hot Leg 2nd Isolation Valve	1.0
8.	1204-U4-123	2	SI-Hot Leg 2nd Isolation Valve	1.0
9.	1204-U6-079	10	Accumulator 2nd Isolation Valve	5.0
10.	1204-U6-080	10	Accumulator 2nd Isolation Valve	5.0
11.	1204-U6-081	10	Accumulator 2nd Isolation Valve	5.0
12.	1204-U6-082	10	Accumulator 2nd Isolation Valve	5.0
13.	1204-U6-083	10	Injection Line 1st Isolation Valve	5.0
14.	1204-U6-084	10	Injection Line 1st Isolation Valve	5.0
15.	1204-U6-085	10	Injection Line 1st Isolation Valve	5.0

TABLE 5.4.12-3 (SHEET 2 OF 2)

REACTOR COOLANT SYSTEM PRESSURE ISOLATION VALVES

	VALVE <u>NUMBER</u>	VALVE <u>SIZE (in.)</u>	<u>FUNCTION</u>	MAXIMUM ALLOWABLE <u>LEAKAGE(gpm)</u>
16.	1204-U6-086	10	Injection Line 1st Isolation Valve	5.0
17.	1204-U6-124	6	SI-Hot Leg 1st Isolation Valve	3.0
18.	1204-U6-125	6	SI-Hot Leg 1st Isolation Valve	3.0
19.	1204-U6-126	6	SI-Hot Leg 1st Isolation Valve	3.0
20.	1204-U6-127	6	SI-Hot Leg 1st Isolation Valve	3.0
21.	1204-U6-128	8	RHR-Hot Leg 2nd Isolation Valve	4.0
22.	1204-U6-129	8	RHR-Hot Leg 2nd Isolation Valve	4.0
23.	1204-U4-143	2	SI-Cold Leg 2nd Isolation Valve	1.0
24.	1204-U4-144	2	SI-Cold Leg 2nd Isolation Valve	1.0
25.	1204-U4-145	2	SI-Cold Leg 2nd Isolation Valve	1.0
26.	1204-U4-146	2	SI-Cold Leg 2nd Isolation Valve	1.0
27.	1204-U6-147	6	RHR Cold Leg 2nd Isolation Valve	3.0
28.	1204-U6-148	6	RHR Cold Leg 2nd Isolation Valve	3.0
29.	1204-U6-149	6	RHR Cold Leg 2nd Isolation Valve	3.0
30.	1204-U6-150	6	RHR Cold Leg 2nd Isolation Valve	3.0

TABLE 5.4.13-1

PRESSURIZER SAFETY AND RELIEF VALVES DESIGN PARAMETERS

Pressurizer safety valves

Number 3

Minimum relieving capacity at 2560 psig per 420,000

valve, ASME flowrate (lb/h)

Set pressure (psig) 2460 Design temperature (°F) 650

Fluid Saturated steam

Backpressure

Normal (psig) 3 to 5 Expected maximum during discharge (psig) 500

Environmental conditions

Ambient temperature (°F) 50 to 120 Relative humidity (percent) 0 to 100

Pressurizer power-operated relief valves

Number 2
Design pressure (psig) 2485
Design temperature (°F) 650
Saturated steam-relieving capacity at 2385 psig, 210,000

per valve (lb/h)

Saturated water-relieving capacity at 2485 psig, 230

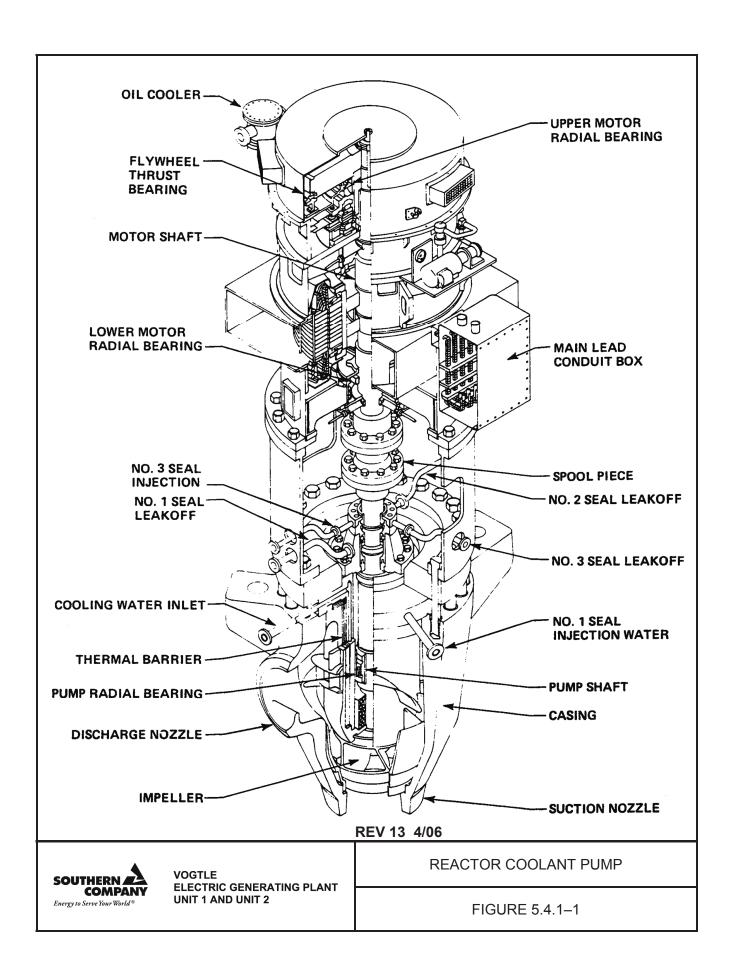
per valve (gal/min)

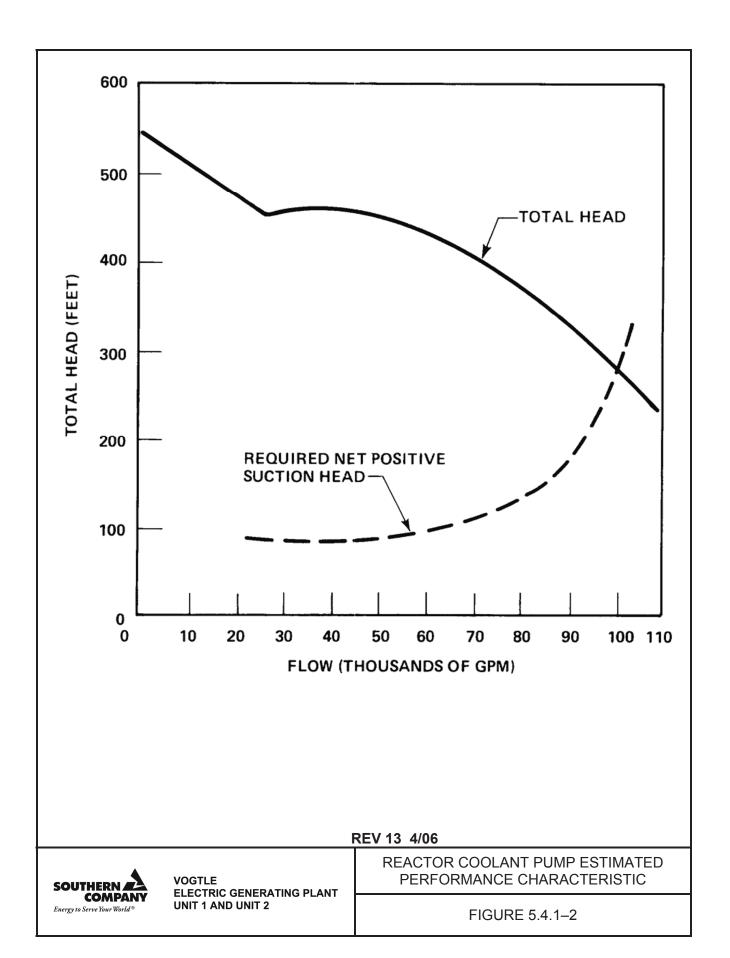
TABLE 5.4.15-1

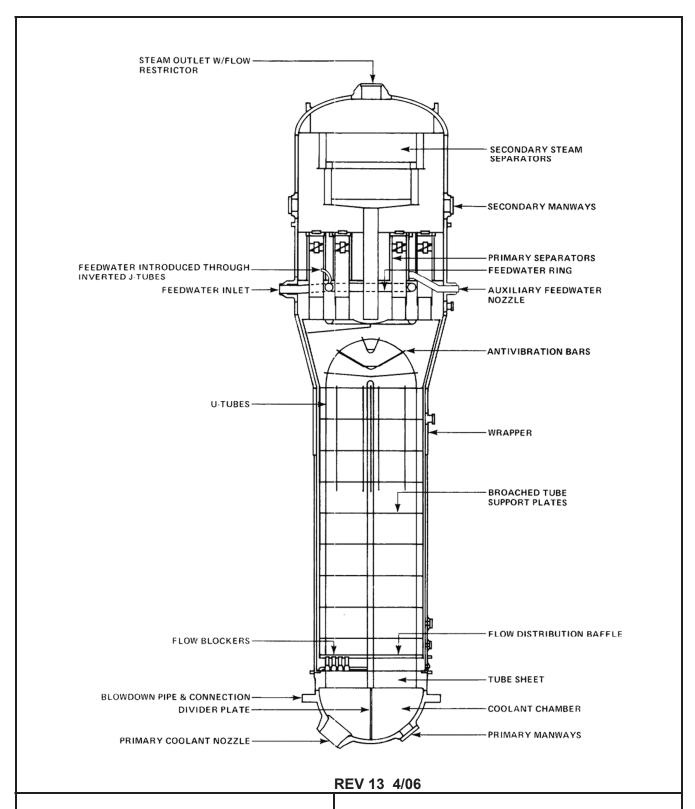
REACTOR VESSEL HEAD VENT SYSTEM EQUIPMENT DESIGN PARAMETERS

<u>Valves</u>

Number (includes one manual valve) Design pressure, psig Design temperatures, °F	7 2485 650
Piping	
Vent line, nominal diameter, in. Design pressure, psig Design temperature, °F Maximum operating temperature	1 2485 650 620



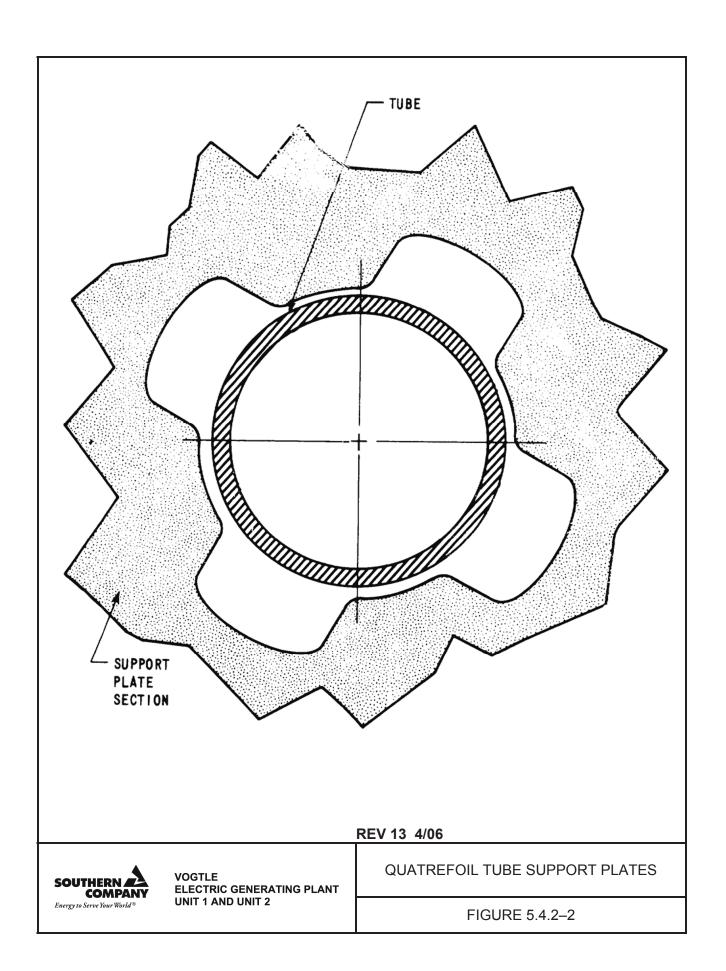


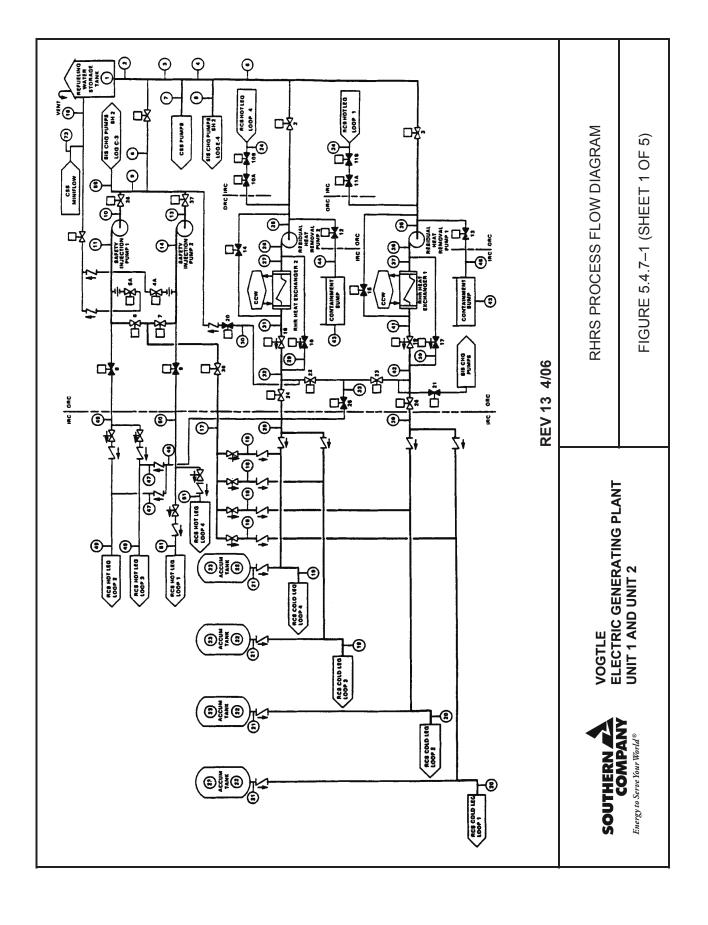


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VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 MODEL F STEAM GENERATOR

FIGURE 5.4.2-1





MODES OF OPERATION

MODE A INITIATION OF RHR OPERATION

When the reactor coolant temperature and pressure are reduced to 350°F and 365 psig, approximately 2 to 4 hours after reactor shutdown, the second phase of plant cooldown starts with the RHRS being placed in operation. Before starting the pumps, the inlet isolation valves are opened, the heat exchanger flow control valves are set at minimum flow, and the outlet valves are verified open. The automatic miniflow valves are open and remain so until the pump flow exceeds the closed setpoint (1944 gpm at 350°F, 1841 gpm at 100°F), at which time they trip closed. Should the pump flow drop below the open setpoint (824 gpm at 350°F, 780 gpm at 100°F), the miniflow valves open automatically.

Startup of the RHRS includes a warmup period, during which reactor coolant flow through the heat exchangers is limited to minimize thermal shock on the RCS. The rate of heat removal from the reactor coolant is controlled manually by regulating the reactor coolant flow through the residual heat exchangers. The total flow is regulated automatically by control valves in the heat exchanger bypass line to maintain a constant total flow. The cooldown rate is limited to 100°F/h based on equipment stress limits and a 120°F maximum component cooling water temperature.

MODE B END CONDITIONS OF A NORMAL COOLDOWN

This situation characterizes most of the RHRS operation. As the reactor coolant temperature decreases, the flow through the residual heat exchanger is increased until all of the flow is directed through the heat exchanger to obtain maximum cooling.

NOTE

For the safeguards functions performed by the RHRS, refer to section 6.3.

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VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 RHRS PROCESS FLOW DIAGRAM

FIGURE 5.4.7–1 (SHEET 2 OF 5)

VALVE ALIGNMENT CHART

Vo.1	El en Diamen	Operati	onal Mode (a)
Valve No.	Flow Diagram Valve No.	<u>A</u>	<u>B</u>
HV-8812B	2	С	С
HV-8812A	3	С	C
HV-8702A/B	10	0	0
HV-8701A/B	11	0	O
HV-8811B	12	С	C
HV-8811A	13	С	Ċ
FV-0611	14	С	Ċ
FV-0610	15	С	C
FV-0619	16	P	С
FV-0618	17	P	C
HV-0607	18	P	P
HV-0606	19	P	P
HV-8804B	20	С	C
HV-8804A	21	С	C
HV-8716B	22	С	Ċ
HV-8716A	23	С	C
HV-8809B	24	0	0
HV-8809A	26	0	0

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VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2

RHRS PROCESS FLOW DIAGRAM

FIGURE 5.4.7–1 (SHEET 3 OF 5)

a. O = Open. C = Closed.

P = Partially open.

MODE A INITIATION OF RHR OPERATION

	$(1b/h \times 10^6)$	1 350	0.50	1.340	1.340	0.740	0.740	0.600	1,340	1.340	0.680	0.660	1.340	1.340	1.340	0.740	0.740	0.600	1.340	1.340	0.664	0.676	
Flow	(gal/min)	3000	2000	3000	3000	1657	1657	1343	3000	3000	1552	1478	3000	3000	3000	1657	1657	1343	3000	3000	1487	1513	
	Temperature (°F)	350	200	350	350	350	140	350	234	234	234	234	350	350	350	350	140	350	234	234	234	234	
	Pressure (psiq)	007	202	417	565	541	536	492	492	432	400	400	400	418	567	543	536	492	492	435	400	400	
	Fluid	S.	2	SC	RC	SC.	RC	RC	RC	RC	SC C	RC	RC	RC	RC	RC	RC	RC	RC	RC	RC	RC	
	Location	940	+2	25	26	27	31	29	32	28	19 Loop 4	19 Loop 3	34	35	36	37	41	39	42	38	Loop	20 Loop 2	

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RHRS PROCESS FLOW DIAGRAM

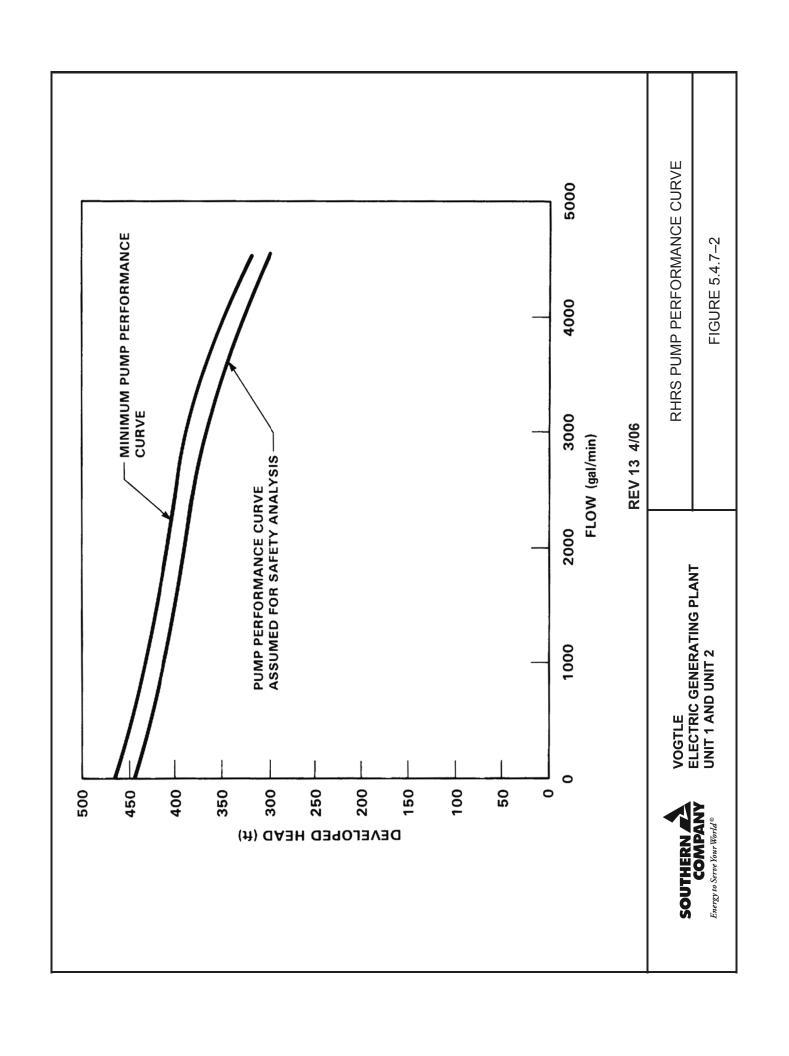
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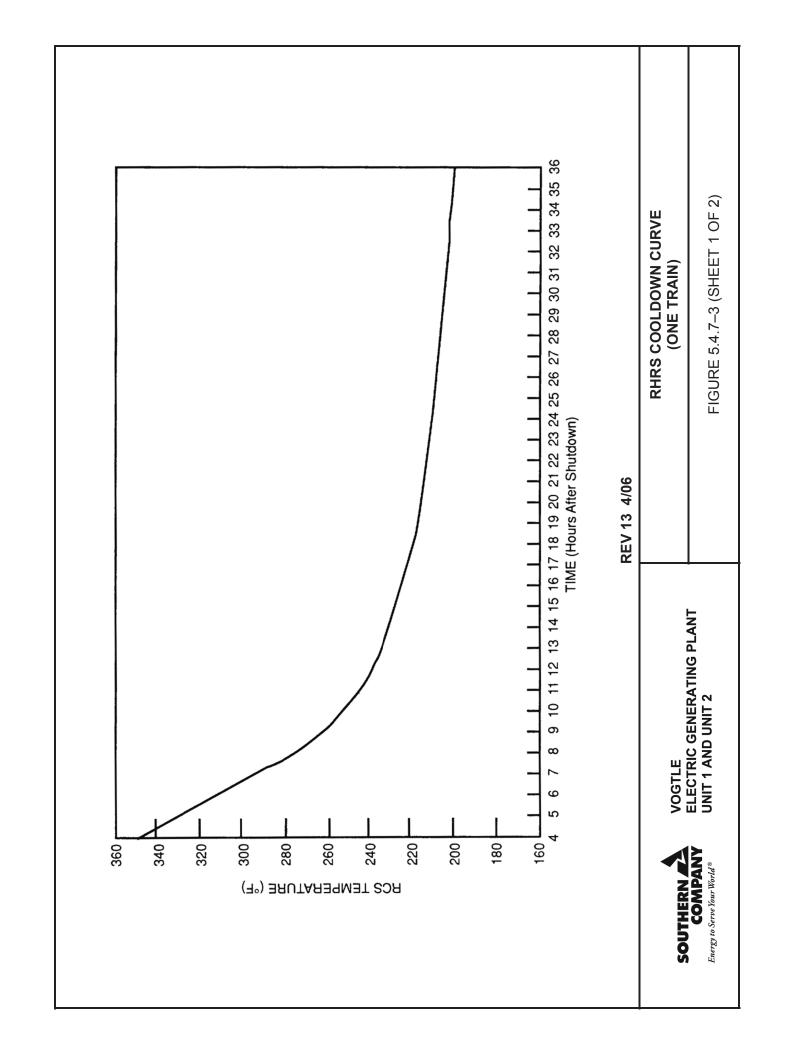
VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2

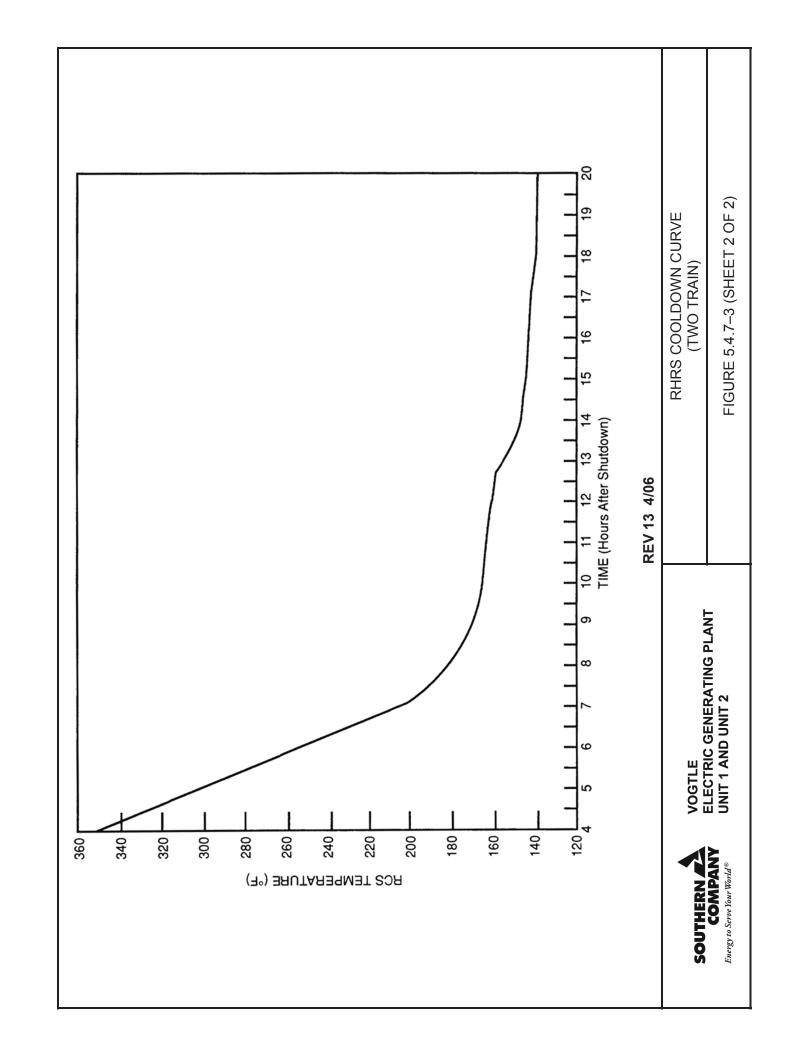
FIGURE 5.4.7-1 (SHEET 4 OF 5)

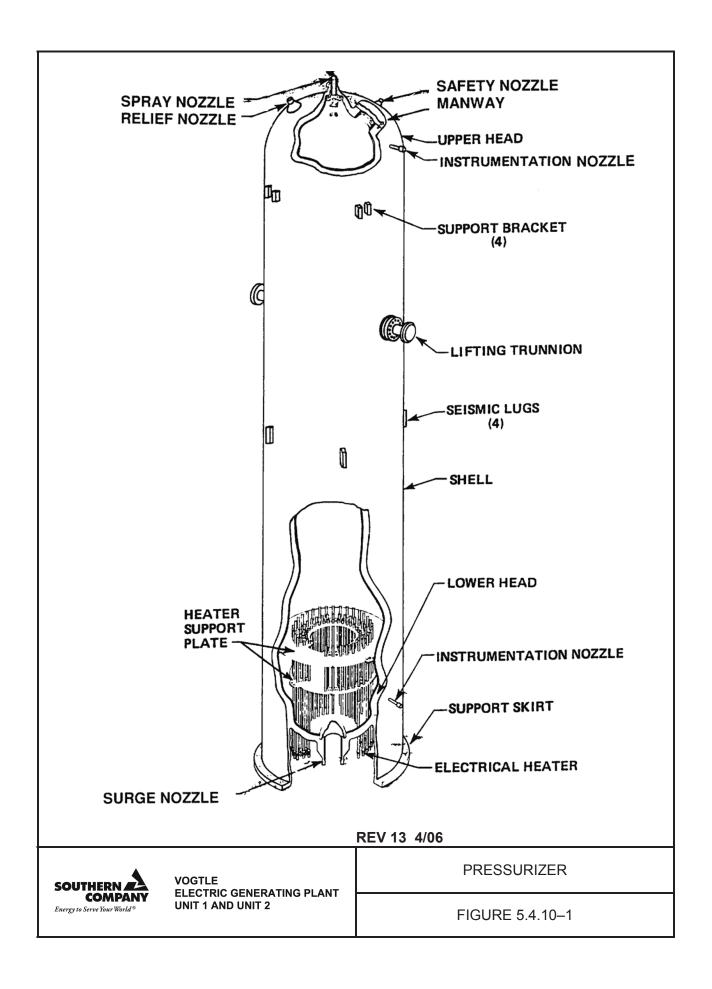
At reference conditions 350°F and 400 psig.

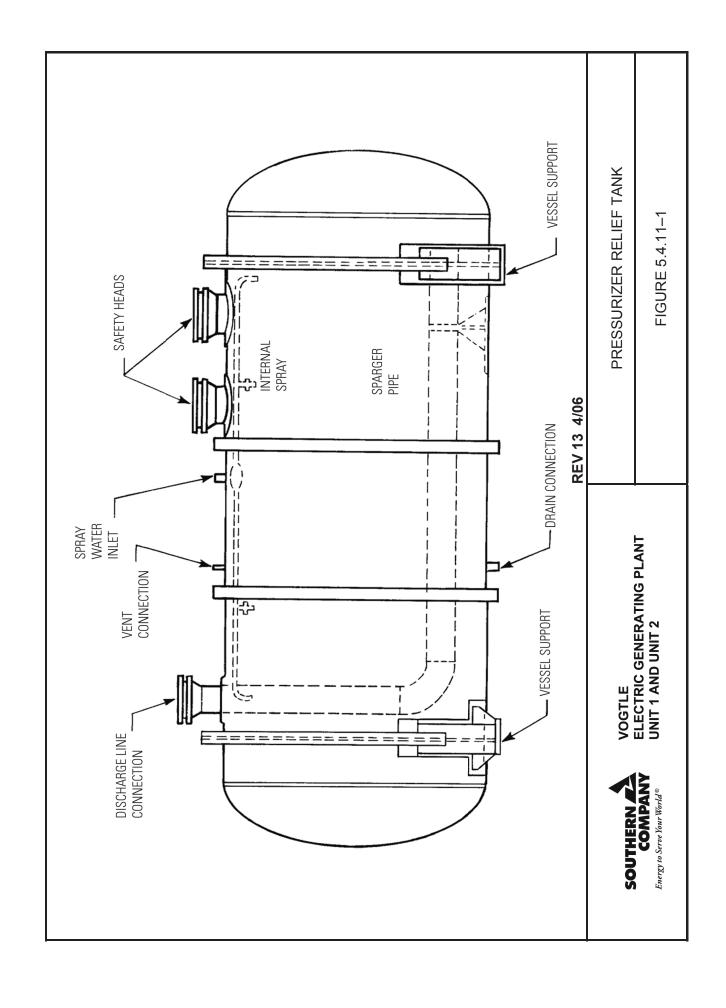
ET 5 OF 5)	FIGURE 5.4.7–1 (SHEET		VIT 2	UNIT 1 AND UNIT	COMPANY Energy to Serve Your World®
/ DIAGRAM	RHRS PROCESS FLOW DIAGRAM	<u>«</u>	VOGTLE	VOGTLE	SOUTHERN
		REV 13 4/06			
0 1.480 1.480 0.733 0.747	3000 3000 1486 1514	55 55 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	0 0 0 0	22222	29 42 38 20 Loop 1 20 Loop 2
1.480 1.480 1.480 1.480	3000 3000 3000 3000	140 140 140 115	19 184 154 136	\$\$\$\$\$\$ \$	34 34 35 36 37 41
0 1.480 1.480 0.750	3000 3000 1570	211 211 211 311	105 105 38 0	2222	
1.480 1.480 1.480 1.480	3000 3000 3000 3000	140 140 115 115	0 19 181 152 134	22222	25 27 27 33 33 33 34
(1b/h x 10°)	Flow (gal/min) (a)	Temperature (°F)	Pressure (psig)_	Eluig	Location
	IL COOLDOWN	ODE B END CONDITIONS OF A NORMAL COOLDOWN	MODE B END COND		

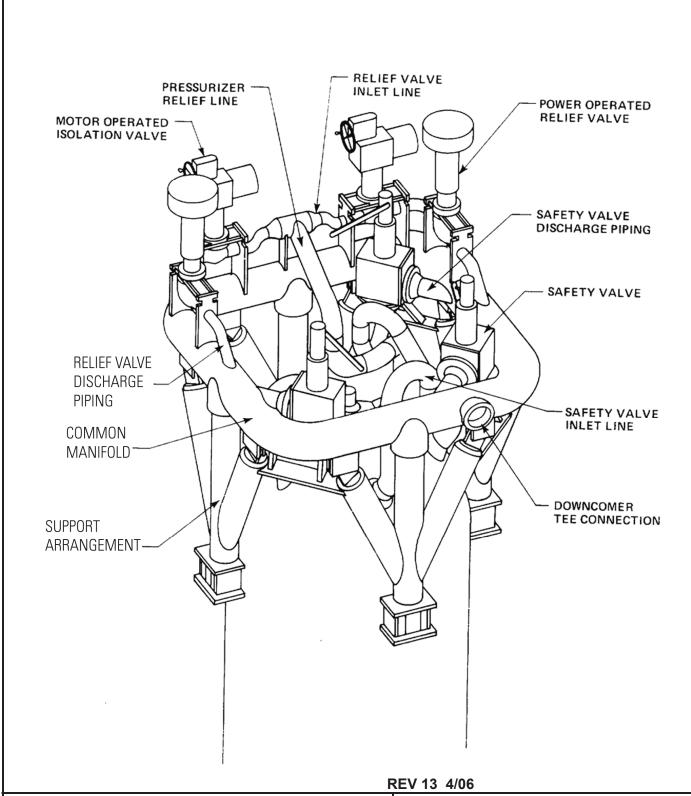








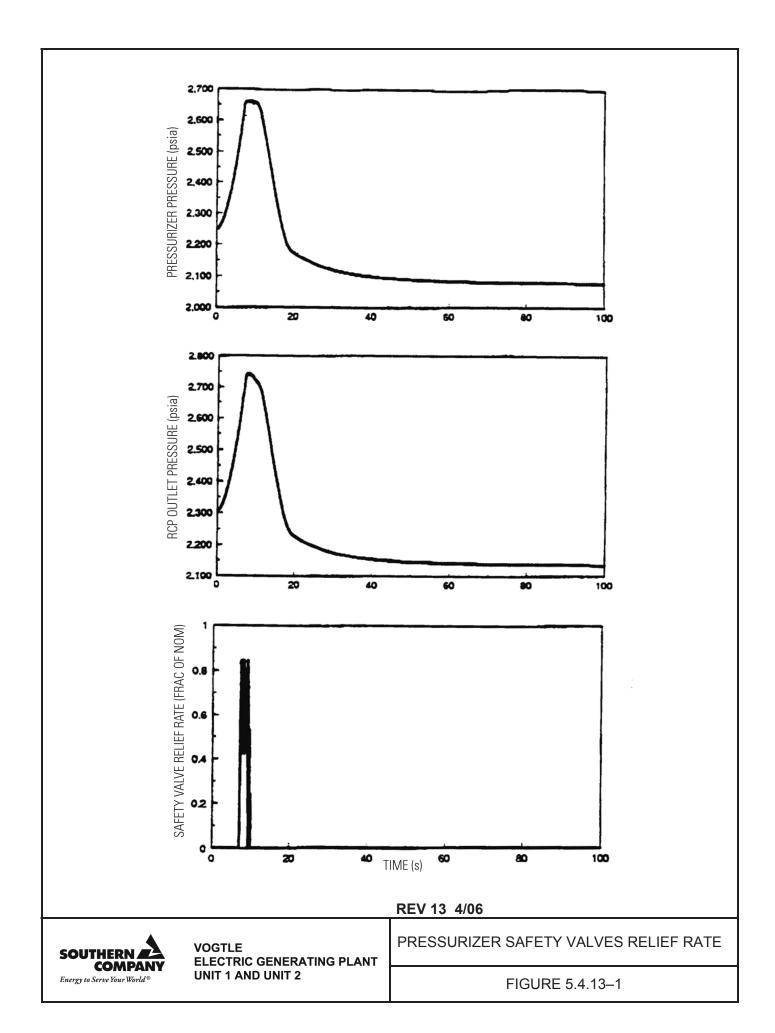


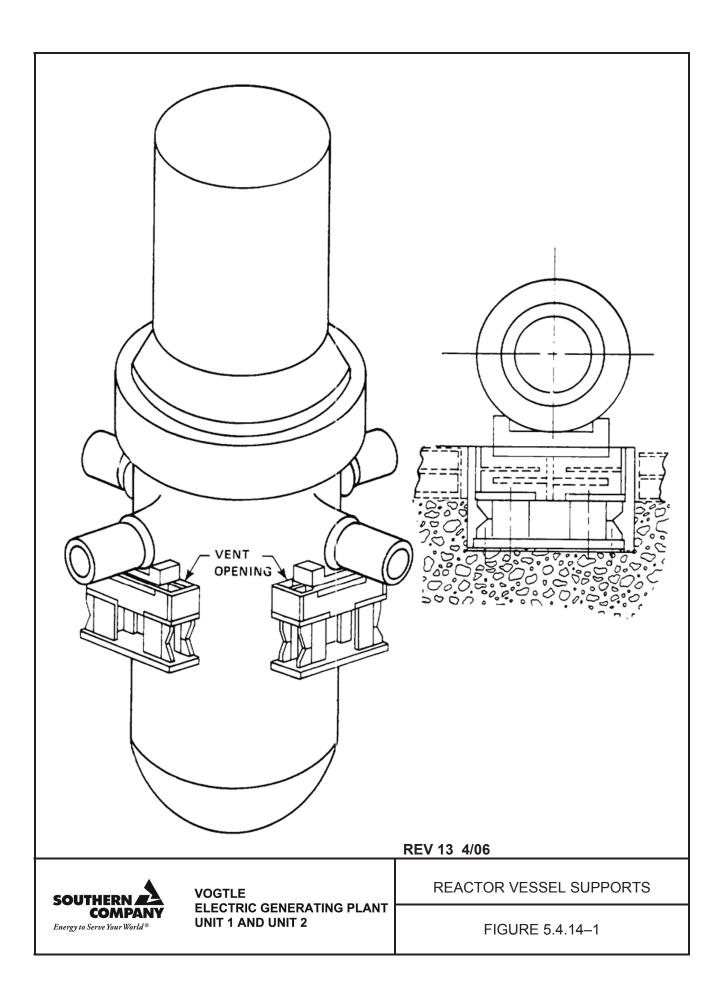


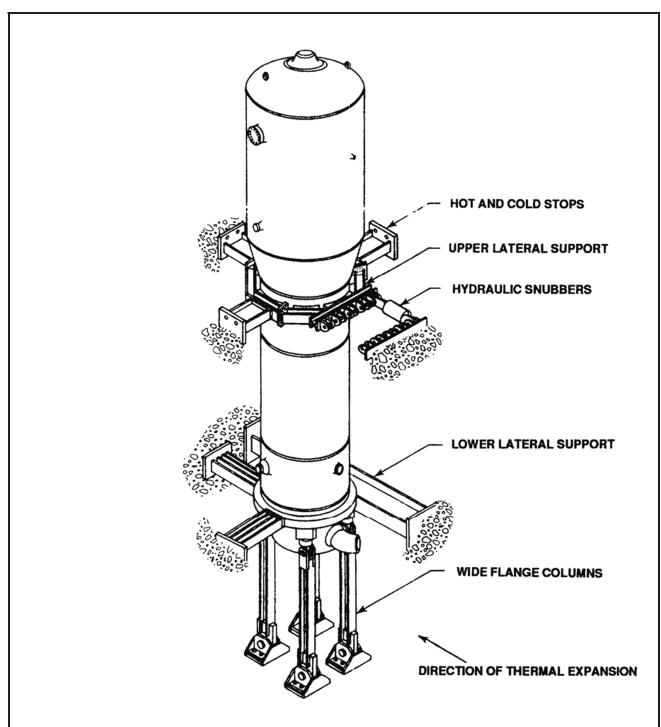


VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 PRESSURIZER SAFETY AND RELIEF VALVE PIPING AND SUPPORT ARRANGEMENT

FIGURE 5.4.11-2







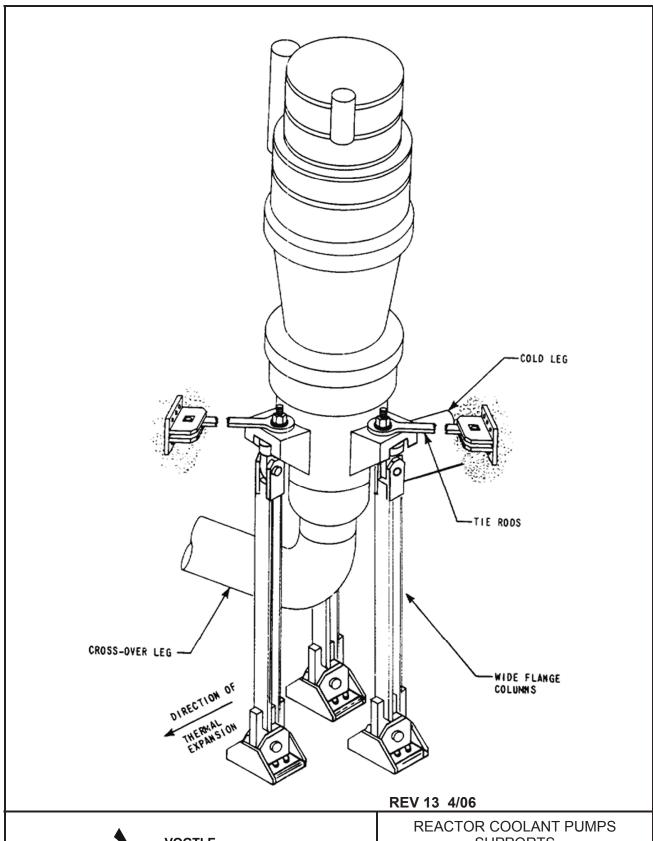
*SYMMETRICALLY INSTALLED WITH RESPECT TO THE CENTER LINE.

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VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 STEAM GENERATOR SUPPORTS

FIGURE 5.4.14-2

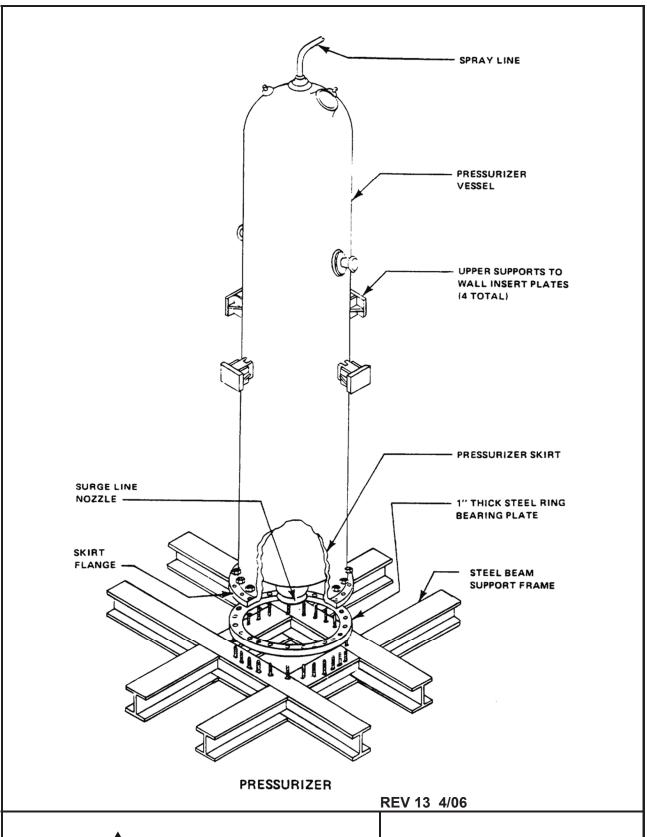




VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2

SUPPORTS

FIGURE 5.4.14-3





VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 PRESSURIZER SUPPORTS

FIGURE 5.4.14-4

6.0 ENGINEERED SAFETY FEATURES

Engineered safety features (ESF) systems protect the public in the event of an accidental release of radioactive fission products from the reactor coolant system (RCS), particularly as the result of a loss-of-coolant accident (LOCA). The safety features function to localize, control, mitigate, and terminate such accidents and to maintain radiation exposure levels to the public below applicable limits and guidelines (e.g., 10 CFR 100). The following systems are defined as ESF systems:

A. Containment Building (subsection 6.2.1)

The containment building is a steel-lined, reinforced, prestressed concrete cylinder with a hemispherical dome and flat circular basemat. The building houses the nuclear steam supply system (NSSS) and is designed to minimize radioactive fission product release from the NSSS to the environs subsequent to postulated design basis accidents (DBAs).

B. Containment Spray System-Iodine Removal System (subsections 6.2.2 and 6.5.2)

The containment spray system provides borated water spray for post-accident containment heat removal and pressure reduction. The iodine removal system provides chemical additives to the containment spray and/or recirculation system for post-accident iodine removal from the containment atmosphere.

C. Containment Fan Cooler System (subsection 6.2.2)

The containment fan cooler system consists of redundant fan cooler units provided for post-accident containment atmosphere heat removal and pressure reduction.

D. Containment Isolation System (subsection 6.2.4)

The containment isolation system provides for automatic containment isolation upon receipt of a containment isolation actuation signal. This system precludes the release of the containment atmosphere to the plant and the surroundings.

E. Combustible Gas Control System (subsection 6.2.5)

The combustible gas control system consists of two hydrogen recombiner subsystems and two hydrogen monitoring subsystems. A post-LOCA cavity purge system is designed to prevent hydrogen pocketing in the reactor cavity. A manually initiated, non-ESF hydrogen purge subsystem is provided as a backup to the ESF subsystems. The system functions to maintain post-LOCA hydrogen concentrations below the combustible limit.

F. The Emergency Core Cooling System (ECCS) (section 6.3)

The ECCS described in section 6.3 injects borated water into the RCS. This provides post-accident cooling of the core to limit core damage and fission product release and ensures adequate shutdown margin. The system also provides continuous long-term, post-accident cooling of the core by recirculation of borated water from the containment sump through the residual heat exchanger and back to the reactor core.

G. Habitability Systems (section 6.4 and subsection 9.4.1)

The control room heating, ventilation, and air-conditioning (HVAC) system is provided to protect control room personnel from post-accident airborne radioactivity.

H. ESF Filter Systems (subsection 6.5.1)

The fuel handling building post-accident exhaust system and the piping penetration filter exhaust system control fission product release resulting from postulated accidents. The control room HVAC system reduces radiation exposures to operating personnel in the control room.

I. Auxiliary Feedwater System (subsection 10.4.9)

The auxiliary feedwater system is provided to automatically supply feedwater to the steam generators for heat removal from the RCS during emergency conditions.

6.1 <u>ENGINEERED SAFETY FEATURES MATERIALS</u>

This section provides a discussion of the materials used in the fabrication of ESF components and of the material interactions that could potentially impair the operation of the ESF.

6.1.1 METALLIC MATERIALS

6.1.1.1 Materials Selection and Fabrication

Information on the selection and fabrication of the materials in the ESF of the plant, such as the ECCS, the containment heat removal systems, the combustible gas control system, and the containment spray systems is provided below. Materials for use in ESF are selected for their compatibility with the reactor coolant system (RCS) and containment spray solutions as described in Section III of the American Society of Mechanical Engineers (ASME) Boiler and Pressure Vessel Code, Articles NC-2160 and NC-3120.

6.1.1.1.1 Specifications for Principal Pressure-Retaining Materials

All pressure-retaining materials in ESF system components comply with the corresponding material specification permitted by ASME Section III, Division 1. The material specifications for pressure-retaining materials in each component of an ESF system meet the requirements of Article NC-2000 of ASME Section III, Class 2, for Quality Group B and Article ND-2000 of ASME Section III, Class 3, for Quality Group C components. Materials produced under American Society of Testing Materials (ASTM) designation are acceptable as complying with the corresponding ASME specification, provided the ASME specification is designated as being identical with the ASTM specification for the grade, class, or type produced and that the material is confirmed as complying with the ASME specification by a certified material test report or certification from the material manufacturer (Subarticle NA/NCA-1220). Containment penetration materials meet the requirements of Articles NC-2000 and NE-2000 of ASME Section III, Division 1. The quality groups assigned to each component are given in table

3.2.2-1. Principal pressure-retaining materials are indicated in table 6.1.1-1. Material specifications for equipment within the NSSS scope are provided in table 5.2.3-1.

6.1.1.1.2 Engineered Safety Features Construction Materials

The welding materials used for joining the ferritic base materials of the ESF conform to or are equivalent to ASME Material Specifications SFA 5.1, 5.2, 5.5, 5.17, 5.18, and 5.20. The welding materials used for joining nickel-chromium-iron alloy in similar base material combination and in dissimilar ferritic or austenitic base material combination conform to ASME Material Specifications SFA 5.11 and 5.14.

The welding materials used for joining the austenitic stainless steel base materials conform to ASME Material Specifications SFA 5.4 and 5.9. These materials are qualified to the requirements of the ASME Code, Section III and Section IX, and are used in procedures which have been qualified to these same rules. The methods utilized to control delta ferrite content in austenitic stainless steel weldments are discussed in section 1.9 and subsection 5.2.3.

Components in contact with borated water are fabricated of or clad with austenitic stainless steel or equivalent corrosion-resistant material. The integrity of the safety-related components of the ESF is maintained during all stages of component manufacture. Austenitic stainless steel is utilized in the final heat-treated condition as required by the respective ASME Code, Section II, material specification for the particular type or grade of alloy. Furthermore, austenitic stainless steel materials used in the ESF components are handled, protected, stored, and cleaned according to recognized and accepted methods which are designed to minimize contamination which could lead to stress corrosion cracking. These controls are stipulated in specifications which are discussed in subsection 5.2.3. Additional information concerning austenitic stainless steel, including the avoidance of sensitization and the prevention of intergranular attack, can be found in subsection 5.2.3. No cold-worked austenitic stainless steels having yield strengths greater than 90,000 psi are used for components of the ESF.

Materials utilized in ESF components within the containment that would be exposed to core cooling water and containment sprays in the event of a LOCA are listed in table 6.1.1-2. These components are manufactured primarily of stainless steel or other corrosion-resistant material. Protective coatings are applied on carbon steel equipment located inside the containment. (See subsection 6.1.2.)

To limit the generation of hydrogen within the containment, restrictions are placed on the use of aluminum, zinc, and mercury in the containment:

- A. Aluminum is severely attacked by the alkaline containment spray solution, which results in the generation of gaseous hydrogen and the possible loss of structural integrity. The amount of aluminum present inside the containment is restricted to an essential minimum.
- B. Boric acid spray reacts with zinc, oxidizing it and liberating hydrogen gas. The use of zinc in the containment is minimized to reduce generation of hydrogen.

Table 6.2.5-6 contains a list of the amounts of aluminum and zinc which are expected to be present in the containment and which could potentially be exposed to a corrosive environment. These materials are listed by the system or component in which they are used, and an estimate of their expected corrosion rate is given. The use of mercury and mercuric compounds is prohibited inside the containment. Temporary use of fluorescent and high-pressure sodium lamps

is permitted during refueling outages/plant shutdowns during Modes 5 and 6 only. Usage during these times is administratively controlled.

6.1.1.1.3 Integrity of Safety-Related Components

The integrity of the materials of construction for ESF equipment when exposed to post-DBA conditions have been evaluated. Post-DBA conditions were conservatively represented by test conditions. The test program⁽¹⁾ considered spray and core cooling solutions of the design chemical compositions, as well as the design chemical compositions contaminated with corrosion and deterioration products which may be transferred to the solution during recirculation. The effects of sodium (free caustic), chlorine (chloride), and fluorine (fluoride) on austenitic stainless steels were considered. Based on the results of this investigation, as well as testing by Oak Ridge National Laboratory and others, the behavior of austenitic stainless steels in the post-DBA environment is acceptable. No cracking is anticipated on any equipment even in the presence of postulated levels of contaminants, provided the core cooling and spray solution Ph is maintained at an adequate level. The inhibitive properties of alkalinity (hydroxyl ion) against chloride cracking and the inhibitive characteristic of boric acid on fluoride cracking have been demonstrated.

The selection, procurement, testing, storage, and installation of all nonmetallic thermal insulation ensures that the leachable concentrations of chloride, fluoride, sodium, and silicate are in conformance with Regulatory Guide 1.36, Nonmetallic Thermal Insulation for Austenitic Stainless Steel.

Conformance with Regulatory Guide 1.36 is summarized in section 1.9.

Information is provided in section 1.9 concerning the degree of conformance with the following Regulatory Guides:

- A. 1.31. Control of Ferrite Content in Stainless Steel Weld Metal.
- 1.36, Nonmetallic Thermal Insulation for Austenitic Stainless Steel.
- C. 1.37, Quality Assurance Requirements for Cleaning of Fluid Systems and Associated Components of Water-Cooled Nuclear Power Plants.
- D. 1.44, Control of the Use of Sensitized Stainless Steel.

6.1.1.2 <u>Composition, Compatibility, and Stability of Containment and Core</u> Spray Coolants

The information given below is provided on the composition, compatibility, and stability of the core cooling water and the containment sprays of the ESF.

Supply for the containment sprays and ECCS is drawn from the refueling water storage tank. As described in sections 3.8 and 6.3, the refueling water storage tank is a stainless steel-lined concrete tank not subject to significant corrosive attack by the tank's contents. Trisodium phosphate for recirculation fluid pH adjustment is stored in baskets located in the containment.

The accumulator tanks, which store boric acid solution (1900-2600 ppm) for the accumulator portion of the safety injection system, are made of carbon steel and clad with stainless steel to ensure corrosion resistance.

The boron injection tank (Unit 1 only) is stainless steel. Because of the corrosion resistance of this material, significant corrosive attack on the vessel is not expected. The boron injection tank contains a boron concentration, as boric acid, of 0-2600 ppm.

6.1.1.3 Reference

1. Picone, L. F., and Whyte, D. D., "Behavior of Austenitic Stainless Steel in Post Hypothetical Loss-of-Coolant Environment," <u>WCAP-7798-L</u> (Proprietary), November 1971, and WCAP-7803 (Nonproprietary), December 1971.

6.1.2 ORGANIC MATERIALS

6.1.2.1 <u>Protective Coatings</u>

Certain coatings, which are in common industrial use, may deteriorate in the post-accident environment and may contribute substantial quantities of foreign solids and residue to the containment sump. Consequently, protective coatings used inside the containment, excluding components limited by size and/or exposed surface area, are demonstrated to withstand the design basis accident (DBA) conditions (subsection 3.11.B.1) and meet the intent of American National Standards Institute (ANSI) N101.2 (I972), Protective Coatings (Paints) for Light Water Nuclear Reactor Containment Facilities, as well as the recommendations of Regulatory Guide 1.54, Quality Assurance Requirements for Protective Coatings Applied to Water-Cooled Nuclear Power Plants. Information regarding conformance with Regulatory Guide 1.54 is provided in table 6.1.2-1 and further conformance information for nuclear steam supply system (NSSS) equipment has been submitted to the Nuclear Regulatory Commission (NRC) for review via reference 1 and accepted via reference 2.

- A. Regulatory Guide 1.54 is imposed for items located within the containment building as follows:
 - 1. For shop priming of liner plate, structural steel, and fabricated shapes.
 - 2. For shop priming of fabricated pipes, tanks, heating, ventilation, and airconditioning (HVAC) ducts, and equipment.
 - 3. For field finish painting of steel where called for in drawings and specifications.
 - 4. For surfacing of concrete where indicated in drawings and specifications.
- B. Regulatory Guide 1.54 is implemented by requirements as follows:
 - 1. Use of specific coatings systems which are pregualified to ANSI N101.2.
 - 2. Surface preparation standards.
 - Surface profile requirements.
 - 4. Application of the coating systems in accordance with instructions approved by the paint manufacturer.
 - 5. Inspections and nondestructive examinations.

- 6. Identification of all nonconformances. Coatings which do not conform with the VEGP position on Regulatory Guide 1.54 are limited in use and are evaluated on a case basis relative to impact on plant safety.
- 7. Certifications of compliance and/or documentation procedures to satisfy project requirements.
- 8. The vendor's procedures are subject to review prior to application, and the vendor's implementation of the specification requirements is monitored.
- 9. An inventory of unqualified coatings is maintained to ensure appropriate control of coatings inside containment.
- C. Regulatory Guide 1.54 is not imposed for the following:
 - 1. Surfaces to be insulated.
 - 2. Surfaces "contained" within a cabinet or enclosure; for example, the interior surfaces of ducts.
 - 3. Field repair to any small areas previously coated with a qualified coating system such as:
 - a. Bolt heads, nuts, and miscellaneous fasteners.
 - b. Damage resulting from spot, tack, or stud welding.

Field touchup and repair of large areas shall be in accordance with Regulatory Guide 1.54.

- 4. Small "production line" items such as small motors, handwheels, pipe supports, snubbers, electrical cabinets, control panels, loudspeakers, etc., where special painting requirements would be impracticable.
- Stainless steel or galvanized surfaces.
- 6. Coating used for the banding of piping.
- 7. Concrete designated to receive a sealer coat only.
- D. The majority of the coatings specified for use inside the containment are the inorganic type (ethyl silicate inorganic zinc). The mode of failure of inorganic zinc is powdering rather than blistering and delamination. This failure mode minimizes the accumulation of solid debris in the containment sumps. Any particles of appreciable size that do occur either settle out prior to reaching the sump screens or are trapped by the sump filter screens. The screen opening size (3/32 in.) for the Emergency Containment Cooling System (ECCS) Residual Heat Removal (RHR) Systems is smaller than the line piping, the RHR heat exchanger tubes, the branch line needle valves (used for throttling), pump running clearances, and clearances in the reactor core so particles that could potentially block the system are filtered out. (Refer to section 6.2 for a discussion of the sump design and consideration given to screen clogging.)

The screen opening size is 3/32-in. diameter by design for the Containment Spray (CS) System and the 3/32-in. diameter is smaller than the line piping, the spray nozzles, and pump running clearances, so particles that could potentially block the system are filtered out. (Refer to section 6.2 for a discussion of the sump design and consideration given to screen clogging.) After the new screens were installed, it was discovered that 124 holes (0.002% of total screen surface

area) in the plates had a hole > 3/32-in. diameter. No round holes were found to be > $\frac{1}{4}$ -in. diameter in the screens and all hole areas had < $\frac{1}{4}$ -in. length in the major axis through the minor axis. The CS nozzles have a $\frac{3}{8}$ -in. diameter hole; therefore, blockage in the system will not occur.

E. "N" AREAS

Areas within the containment are identified as "N" areas. Coatings used inside the containment, where required, are the prequalified coating systems. These coatings are prequalified to the intent of ANSI N101.2 and applicable portions of ANSI N5.12. Quality assurance and documentation requirements of ASME NQA-1-1994, as described in the SNC Quality Assurance Topical Report (QATR) and ANSI N101.4 (Class I) are enforced for both coating materials and applications procedures as discussed in table 6.1.2-1.

F. "D" AREAS

Areas outside the containments, but with potential contamination from radioactive sources, are identified as "D" areas. Coating materials specified for "D" areas are either the same or similar materials which are used in "N" areas, except the quality assurance program and the documentation requirements of ANSI do not apply (Class II).

G. "C" AREAS

Areas outside the containments, and not subject to potential contamination from radioactive sources are identified as "C" areas. "C" area coatings are standard commercial coatings formulated to withstand exposure to industrial environment and require minimum maintenance (Class II).

A coating schedule for items inside the containment is given in tables 6.1.2-2 and 6.1.2-3. Approximate paint film thickness and exposed surface area for major components and structures inside the containment are also provided. The painted areas of valve operators, miscellaneous parts on the reactor coolant pump drives, and instrumentation are considered insignificant. Exposed concrete in the containment is coated as indicated in table 6.1.2-2. The containment temperature profile used for the specification of coatings used inside containment is shown in figure 6.1.2-1.

Protective coatings for use on NSSS components in the reactor containment have been evaluated as to their suitability in post-DBA conditions. Tests have shown that the inorganic zinc, epoxy, and modified phenolic systems are the most desirable of the generic types evaluated for use inside containment. This evaluation⁽³⁾ considers resistance to high temperature and chemical conditions anticipated during a loss-of-coolant accident, as well as high radiation resistance.

6.1.2.2 Other Organic Materials

A listing of other organic materials in the containment is included in table 6.1.2-4. The materials listed are not protective coatings applied to surfaces of nuclear facilities.

6.1.2.3 References

1. Letter NS-CE-1352, C. Eicheldinger (Westinghouse) to C. J. Heltemes, Jr. (NRC), dated February 1, 1977.

- 2. Letter, C. J. Heltemes, Jr. (NRC) to C. Eicheldinger (Westinghouse), dated April 27, 1977.
- 3. Picone, L. F., "Evaluation of Protective Coatings for use in Reactor Containment," <u>WCAP-7198-L</u> (Proprietary), April 1968, and <u>WCAP-7825</u> (Nonproprietary), December 1971.

6.1.3 POST-ACCIDENT CHEMISTRY

Following a main steam line break or design basis loss-of-coolant accident, trisodium phosphate and boric acid solutions will be present in the containment sumps. Subsection 6.5.2 indicates the quantities of trisodium phosphate and boric acid that will be present in the containment after an accident. The pH control reduces the probability of chloride stress corrosion cracking of stainless steel and maximizes iodine retention in the sump solution.

TABLE 6.1.1-1 (SHEET 1 OF 2)

PRINCIPAL ESF PRESSURE-RETAINING MATERIALS

Component	<u>Material</u>
Piping/tubing	SA-53 Gr. B SA-106 Gr. B and C SA-155 Gr. 70 Class 1 and Gr. KC70 Class 1 SA-213, TP 304, 304L and 316 SA-249, TP 304L SA-312, TP 304 and 304L SA-333 Gr. 1 and 6 SA-335 Gr. P11 and P22 SA-376, TP 304 and 316 SB-111 Gr. CDA 706 SB-466 Gr. CDA 706
Fittings/flanges	SA-105 N SA-181 Gr. I and II SA-182, TP F304, F304L, F316, F316L SA-234 Gr. WPB, WPC, WPBW, and WPCW SA-403 WP 304, 304L, 304W, and 304LW SA-420 Gr. WPL6 SA-479, TP 304, 304L and 316
Plate	SA-240, TP 304, 304L and 316L SA-283 Gr. C SA-285 Gr. A and C SA-515 Gr. 70 SA-516 Gr. 70 SA-537 Class 1 SB-171 Gr. CDA 706
Bolting/nuts/studs	SA-193 Gr. B6, B7, B8, and B8M SA-194, Gr 2H, 4, 6, 7, 8H, 8M, and B8 SA-307 Gr. B SA-320 Gr. L7 SA-453 Gr. 660A and 660B SA-564 Gr. 630

TABLE 6.1.1-1 (SHEET 2 OF 2)

Component	<u>Material</u>
Castings	SA-216 Gr. WCB and WCC SA-217 Gr. WC9 SA-351 Gr. CF8M and CF3M SA-487 Gr. CA6NM SB-61 SB-62 Gr. CDA 836 SB-148 Gr. CA 952 ASTM-A276 TP 410
Forgings	SA-105 SA-182, TP F304, F304L, F316, and F316L; Gr. F11 and F22 SA-240 TP 304 and 316 SA-350 Gr. LF1 and LF2 SA-479, TP 304, 304L and 316
Bars	SA-479, TP 304, 316 and 410 Gr 316L and F316 SA-564 Gr. 630
Weld rod	SFA 5.1, E 6010 and E 7018 SFA 5.4, E 308-16, E 308L-16 and E 309 SFA 5.9, ER 308, ER 308L, and ER 309 SFA 5.17, EM 12K SFA 5.18, E 70S-2, E 70S-3, E 70S-4, E70S-6, and E70S-1B SFA 5.20, E 70T-1 and 70T-5

TABLE 6.1.1-2 (SHEET 1 OF 2)

PRINCIPAL ESF MATERIALS EXPOSED TO REACTOR COOLANT OR CONTAINMENT SPRAY

<u>Component</u> <u>Material</u>

Piping/tubing SA-106 Gr. B and C

SA-155 Gr. KC70 Class 1 and 70 Class 1

SA-213, TP 304, 304L, and 316

SA-249, TP 304L

SA-312, TP 304 and 304L SA-333 Gr. 1 and 6 SA-376, TP 304 and 316 SB-111 Gr. CDA 706 SB-466 Gr. CDA 706

Fittings/flanges SA-105 N

SA-181 Gr. I and II

SA-182, TP F304, F304L, F316, and F316L SA-234 Gr. WPB, WPBW, WPCW, and WPC SA-403, WP 304, 304L, 304W, and 304LW

SA-420 Gr. WPL6

SA-479, TP 304, 304L, and 316

Plate SA-240, TP 304, 304L, and 316L

SA-285 Gr. A and C SA-515 Gr. 70 SA-516 Gr. 70 SA-537 Class 1 SB-171 Gr. CDA 706 ASTM A 515 Gr. 70

Shapes SA-36

ASTM A-36

ASTM A-500 Gr. B (Code Case N-71-10)

Bolts/nuts/studs/pins SA-193 Gr. B6, B7, B8 and B8M

SA-194 Gr. 2H, 8H, 8M, 7, 4, 6 and B8

SA-307 Gr. B SA-320 Gr. L7 SA-325 Type 1

SA-453 Gr. 660A and 660B

TABLE 6.1.1-2 (SHEET 2 OF 2)

<u>Component</u> <u>Material</u>

SA-564 Gr. 630 ASTM A 193 Gr. B7 ASTM A 194 Gr. 7 ASTM A 307 ASTM A 354 ASTM A 490

Bars SA-479, TP 304, 316, 410, 316L, and F316

SA-564 Gr. 630

ASTM A 108 Gr. 1018 CW (Code Case N-71-5)

Forgings SA-105

SA-182, TP F304, F304L, F316 and F316L; Gr F11

and F22

SA-240, TP 304 and 316 SA-350 Gr. LF1 and LF2

SA-479, TP 304, 304L and 316

ASTM A 668 Class C (Code Case N-71-5)

Castings SA-216 Gr. WCB and WCC

SA-217 Gr. WC9

SA-351 Gr. CF8M and CF3M

SA-487 Gr. CA6NM

SB-61

SB-62 Gr. CDA 836 SB-148 Gr. CA 952 ASTM A 276 TP 410 ASTM A-216 Gr. WCB

Cooling coil fins SB-152 Gr. CDA 122

TABLE 6.1.2-1 (SHEET 1 OF 3)

REGULATORY GUIDE 1.54, REVISION 0, JUNE 1973, QUALITY ASSURANCE REQUIREMENTS FOR PROTECTIVE COATINGS APPLIED TO WATER-COOLED NUCLEAR POWER PLANTS

Regulatory Guide 1.54 Position

Position on Non-NSSS Components

The requirements and guidelines included in ANSI N101.4-1972, Quality Assurance for Protective Coatings Applied to Nuclear Facilities, for protective coatings applied to ferritic steels, aluminum, stainless steel, zincoated (galvanized) steel, concrete, or masonry surfaces of water-cooled nuclear power plants are generally acceptable and provide an adequate basis for complying with the pertinent quality assurance requirements of Appendix B to 10 CFR 50 subject to the following:

 ANSI N101.4-1972 should be used in conjunction with ASME NQA-1-1994, Quality Assurance Requirements for Nuclear Facility Applications, as described in the SNC QATR.

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The requirements of VEGP are that practical and adequate corrosion protection shall be provided for all surfaces being painted. All surfaces, however, are not coated with coating materials tested and accepted under ANSI N101.2 or ANSI N5.12 criteria, nor are all coatings documented as outlined under ANSI N101.4-1972.

Coating materials used for items located outside of the containment are not documented, since removal of the coating from such items does not affect the safe shutdown of the facility.

Coating materials used for items located within the containment shall meet, whenever possible, the requirements of ANSI N101.2 and selected portions of ANSI N5.12 and are documented in accordance with ANSI N101.4.

Position on NSSS Components

Westinghouse has developed an alternate approach to ANSI N101.4 for satisfying Reg. Guide 1.54 for the NSSS components inside containment. Stringent requirements are specified for the painting of major components in Westinghouse Process Specifications that are imposed on vendors by procurement documents. Large equipment must have coating systems qualified to meet ANSI N101.2 and requirements are defined for surface preparation, use of undercoating, and where applicable, inspection. Other major equipment is either fabricated from stainless steel or covered by insulation. For small items of equipment, conventional industry practices are applied. Details of this approach follow:

NSSS equipment located in the containment building is separated into four categories to identify the applicability of this regulatory guide to various types of equipment. These categories of equipment are as follows:

Category 1 - Large equipment
Category 2 - Intermediate equipment
Category 3 - Small equipment
Category 4 - Insulated/stainless steel equipment

A discussion of each equipment category follows:

Category 1 - Large Equipment

The Category 1 equipment consists of the following:

- Reactor coolant system supports.
- Reactor coolant pumps (motor and motor stand)
 - Accumulator tanks.
- Refueling machine.

Since this equipment has a large surface area and is procured from only a few vendors, it is

TABLE 6.1.2-1 (SHEET 2 OF 3)

Regulatory Guide 1.54 Position

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- when references are made to other standards, these will be covered separately in other regulatory guides applicability or acceptability of referenced standards editions of the referenced standards. The specific Subdivision 2.7 of ANSI N101.4-1972, states that references shall imply the most recent or current where appropriate.
- service. This statement should not be interpreted as actions is the production of specified documentation. comprises all those planned and systematic actions adequate confidence that shop or field coating work those planned and systematic actions necessary to Subdivision 1.1.2 of ANSI N101.4-1972 states that N101.4-1972 should be considered to comprise all necessary to provide specified documentation and implying that the end product of quality assurance quality assurance, as covered by this standard, for nuclear facilities will perform satisfactorily in provide adequate confidence that shop or field The term "quality assurance" as used in ANSI coating work for nuclear facilities will perform satisfactorily in service.

n this connection it is emphasized that records and and included in the standard, are suggested forms only. Alternate documentation consistent with the documents listed in subdivisions 7.4 through 7.8, requirements of Appendix B to 10 CFR 50 is also considered acceptable.

Position on Non-NSSS Components

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- VEGP follows Steel Structures Painting Council (SSPC-1963 and 1971), ANSI N101.2-1972, and sections, where applicable, of ANSI N5.12-1974
- also used to ensure the coating work is Alternate means of documentation are Conform. Forms used by the project are similar to those of ANSI N101.4. performed satisfactorily in service.

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adequate confidence that shop or field perform satisfactorily in service are as coating work for nuclear facilities will The planned and systematic actions which are necessary to provide follows.

Regulatory Guide 1.54 is implemented as follows:

- systems which are prequalified to The use of specific coatings ANSI N101.2 are specified. .
- Surface preparation standards are used which illustrate the surface preparation procedures used in ANSI N101.2. αi
 - Surface profile requirements are met. က
- Application of the coating systems instructions approved by the paint are made in accordance with manufacturer 4
- Inspections and nondestructive testing are performed. Ġ.
- Nonconformances are identified and evaluated as discussed in paragraph 6.1.2.1.B. ø.
- Certifications of compliance and/or documentation procedures are furnished to satisfy project requirements ۲.

Position on NSSS Components

possible to implement tight controls over these items. Stringent requirements are specified for protective painting specification in the procurement documents. coatings on this equipment through the use of a This specification defines requirements for:

- Preparation of vendor procedures.
- Use of specific coatings systems which are qualified to ANSI 1. 4.
 - Surface preparation.
- Application of the coating systems in accordance with the paint manufacturer's instructions. ω 4_.
 - Inspections and nondestructive examinations. Exclusive of certain materials. 6. 0. 7. 8.
 - Identification of all nonconformance.
 - Certifications of compliance.

requirements is monitored during quality assurance surveillance personnel, and the vendor's implementation of the specification The vendor's procedures are subject to review by engineering activities. This system of controls provides assurance that the protective coatings will properly adhere to the base metal during prolonged exposure to a post-accident environment present within the containment building.

The Category 2 equipment consists of the following:

- Seismic platform and tie rods.
 - Reactor internals lifting rig.
 - Head lifting rig.

individually have very small surface areas, it is not practical to enforce Since these items are procured from a large number of vendors, and Category 1 items. Another painting specification is used in these the complete set of stringent requirements which are applied to procurement documents. This specification defines to the vendors the requirements for:

- Use of specific coating systems which are qualified to ANSI N101.2.
 - Surface preparation.
- Application of the coating systems in accordance with the paint manufacturer's instructions. તાં છ

TABLE 6.1.2-1 (SHEET 3 OF 3)

Regulatory Guide 1.54 Position

conditions (e.g., by radiation). This limitation is not intended to prohibit the use of trichlorotrifluoroethane quality assurance requirements for coating materials and surface preparation of substrates. Coatings and chemical compounds are those containing chlorides, cleaning materials used with stainless steel should where such elements are leachable or where they not be compounded from or treated with chemical Sections 3 and 4 of ANSI N101.4-1972 delineate contribute to corrosion, intergranular cracking, or degreasing of austenitic stainless steel, provided could be released by breakdown of the chemical fluorides, lead, zinc, copper, sulfur, or mercury adequate removal is ensured prior to painting. stress corrosion cracking. Examples of such Specification MIL-C-81302b for cleaning or compounds containing elements that could compounds under expected environmental which meets the requirements of Military

Position on Non-NSSS Components Position on NSSS Components

Conform.

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The vendor's compliance with the requirements is also checked during quality assurance surveillance activities in the vendor's plant. These measures of degree of assurance that the protective coatings will control provide a high adhere properly to the base metal and withstand the postulated accident environment within the containment building.

Category 3 - Small Equipment

Category 3 equipment consists of the following:

- Transmitters.
- Alarm horns.
- Small instruments.
 - Valves.
- Heat exchanger supports.

These items are procured from several different vendors and are painted by the vendor in accordance with conventional industry practices. Because the total exposed surface area is very small, Westinghouse does not specify further requirements.

Category 4 - Insulated or Stainless Steel Equipment

Category 4 equipment consists of the following:

- Steam generators covered with wrapped insulation.
- Pressurizer covered with wrapped insulation.
- Reactor pressure vessel covered with rigid reflective insulation.
 - Reactor cooling piping stainless steel.
- Reactor coolant pump casings stainless steel.

Since Category 4 equipment is insulated or is stainless steel, no painted surface areas are exposed within the containment. Therefore, this regulatory guide is not applicable for Category 4 equipment.

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TABLE 6.1.2-2 (SHEET 1 OF 10)

CONTAINMENT COMPONENTS - COATING SCHEDULE

Containment Liner Plate System ^{(a} Dome CS Cylinder shell CS Sumps SS Refueling canal walls SS and bottom Basemat CS Reactor cavity walls CS Reactor cavity walls CS Primary Shield ^(a)	Material ^(a) e System ^(a) CS SS SS CS CS CS CS	0.25 0.25 0.25 0.25 0.25 0.25	Area (ff²)(b) 30,800(1) 68,612(1) 220(1) 8,115(1) 14,423(2) 3,700(1) 787(2)	Surface Area(%) ±5 ±5 ±5 ±5 ±5 ±5	Ist Coat ^(c) Inorganic zinc Inorganic NC NC NC Inorganic zinc Inorganic zinc Inorganic zinc zinc zinc zinc zinc zinc zinc z	(mils of) 1st Coat 1st Coat 2.5 2.5 2.5 2.5 2.5 2.5 2.5 2.5 2.5 2.5	2nd Coat Epoxy- polyamide	Thickness (mils) of 2nd Coat 3.0	3rd Coat	(mils) of 3rd Coat	4th Coat	Thickness (mils) of 4th Coat
Cylinder wall Cylinder wall and nozzles Nozzles Upper cone and nozzles Containment Locks anc Equipment Hatch Spherical head Exposed hatch ring Hatch structure (CBI	Cylinder wall and CS Cylinder wall and CS nozzles Nozzles CS Upper cone and CS nozzles Containment Locks and Hatch ^(a) Equipment Hatch Spherical head CS Exposed hatch ring CS Hatch structure (CBI CS details)	0.25 1.00 0.50 0.38 3.00 0.50	805(1) 1,287(1) 54(1) 71(1) 174(3) 172(3) 3,250(4)	t t t t t t t t t t t t t t t t t t t	Inorganic zinc Inorganic zinc Inorganic zinc zinc Inorganic zinc zinc Inorganic zinc zinc zinc zinc	2 2 2 2 2 2 2 5 5 5 5 5 5						

Exterior refueling canal walls

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TABLE 6.1.2-2 (SHEET 2 OF 10)

Minimum Thickness (mils) of 4th Coat																		
4th Coat																		
Minimum Thickness (mils) of 3rd Coat																		
3rd Coat																		
Minimum Thickness (mils) of 2nd Coat																		
2nd Coat																		
Minimum Thickness (mils of) 1st Coat		2.5	2.5		2.5	2.5												
1st Coat ^(c)		Inorganic	Inorganic	2	Inorganic	Inorganic	2		NC	NC	NC		NC	NC		NC	NC	O N
Tolerance Surface Area(%)		+ + - -	ر 1 را	P	+ 5,	ر بر د	?		±10	±10	±10		±10	±10		±10	±10	±10
Estimated Surface Area (ff²) ^(b)		25(4)	52(1)		25(4)	33(1)			2278(5)	1822(5)	26,700(5)		5920(5)	62,692(5)		14,423(5)	3818(6)	787(6)
Thickness (in.)		0.5	2.0		0.5	2.0			72.0	48.0	45.0		72.0	45.0		126.0	0.96	0.96
<u>Material^(a)</u>		S	CS		CS	CS			Concrete	Concrete	Concrete		Concrete	Concrete		Concrete	Concrete	Concrete
<u>Item</u>	Personnel Lock	Barrel	Sleeve	Escape Lock	Barrel	Sleeve	Containment Concrete ^(a)	Dome	Buttresses to 50° above horizontal	Buttresses from 50° to vertical	Dome area less buttress area	Shell	Buttresses	Shell	Internal Structural Concrete ^(a)	Basemat	Reactor cavity walls	Reactor cavity slab

TABLE 6.1.2-2 (SHEET 3 OF 10)

Minimum Thickness (mils) of 4th Coat	3.0	3.0	3.0		3.0	3.0	3.0	3.0	3.0	3.0					3.0
4th Coat	Epoxy (finish)	Epoxy (finish)	Epoxy (finish)		Epoxy (finish)	Epoxy (finish)	Epoxy	(finish) (finish)	Epoxy (finish)	Epoxy (finish) Epoxy (finish)					Epoxy (finish)
Minimum Thickness (mils) of 3rd Coat	10.0	10.0	10.0		15.0	15.0	10.0	10.0	10.0	10.0					15.0
3rd Coat	Epoxy (surfacer)	Epoxy (surfacer)	Epoxy (surfacer)		Epoxy (surfacer)	Epoxy (surfacer)	Epoxy	(surfacer) Epoxy (surfacer)	Epoxy (surfacer)	Epoxy (surfacer) Epoxy (surfacer)					Epoxy (surfacer)
Minimum Thickness (mils) of 2nd Coat	As needed to fill holes	As needed to fill holes	As needed to fill holes		As needed to fill holes	As needed to fill holes	As needed	As needed to fill holes	As needed to fill holes	As needed to fill holes As needed to fill holes					As needed to fill holes
2nd Coat	Epoxy (filler)	Epoxy (filler)	Epoxy (filler)		Epoxy (filler)	Epoxy (filler)	Epoxy	(filler)	Epoxy (filler)	Epoxy (filler) Epoxy (filler)					Epoxy (filler)
Minimum Thickness (mils of) 1st Coat	0.5	0.5	0.5		0.5	0.5	0.5	0.5	0.5	0.5		0.5	0.5	0.5	0.5
1st Coat ^(c)	Epoxy ^(a) (sealer)	Epoxy ^(a) (sealer)	Epoxy (sealer)	O N	Epoxy (sealer)	Epoxy (sealer)	Epoxy	(sealer) Epoxy (sealer)	Epoxy (sealer)	Epoxy (sealer) Epoxy (sealer)		Ероху	(sealer) Epoxy	(sealer) Epoxy	(sealer) Epoxy (sealer)
Tolerance Surface Area(%)	±10	±10	±10	±10	±10	±10	±10	±10	±10	±10 ±10		±10	±10	±10	±10
Estimated Surface Area (ft²) ^(b)	258(6)	584(6)	(9)2909	1394(6)	13,715(7)	787(1)	2014(8)	830(4)	1849(4)	4394(9) 29,722(4)		1114(9)	3045(9)	1055(9)	229(9)
Thickness (in.)	0.09	84.0	48.0	48.0	33.0	12.0	108.0	24.0	72.0	30.0		30.0	36.0	36.0	48.0
<u>Material^(a)</u>	Concrete	Concrete	Concrete	Concrete	Concrete	Concrete	Concrete	Concrete	Concrete	Concrete		Concrete	Concrete	Concrete	Concrete
<u>Item</u>	5-ft 0-in. thick walls	7-ft 0-in. thick walls	4-ft 0-in. thick walls	Refueling canal slab	Filler slab at 171 ft 9 in.	Reactor cavity filler slab	Primary shield	Cavity access walls	North/south beams at 199 ft	Pressurizer walls Secondary shield walls	Air Shaft Concrete Walls	Walls, Nos. 1, 2, and 3 Below 181 ft	0 in. Above 185 ft	U In. No. 4 airshaft	No. 1 and 2 ceiling slab

TABLE 6.1.2-2 (SHEET 4 OF 10)

Minimum Thickness (mils) of 4th Coat	3.0	9.0	3.0	3.0	3.0		3.0	3.0	3.0	3.0	3.0
4th Coat	Epoxy (finish)	Epoxy (finish)	Epoxy (finish)	Epxoy (finish)	Epoxy (finish)		Epoxy (finish)	Epoxy (finish)	Epoxy (finish)	Epoxy (finish)	Epoxy (finish)
Minimum Thickness (mils) of 3rd Coat	15.0	15.0	15.0	15.0	15.0		10.0	10.0	10.0	10.0	10.0
3rd Coat	Epoxy (surfacer)	Epoxy (surfacer)	Epoxy (surface	Epoxy (surface r)	Epoxy (surface	<u>-</u>	Epoxy (surface r)	Epoxy (surface r)	Epoxy (surface r)	Epoxy (surface r)	Epoxy (surface r)
Minimum Thickness (mils) of 2nd Coat	As needed to fill holes	As needed to fill holes	As needed to fill holes	As needed to fill holes	As needed to fill holes		As needed to fill holes	As needed to fill holes	As needed to fill holes	As needed to fill holes	As needed to fill holes
2nd Coat	Epoxy (filler)	Epoxy (filler)	Epoxy (filler)	Epoxy (filler)	Epoxy (filler)		Epoxy (filler)	Epoxy (filler)	Epoxy (filler)	Epoxy (filler)	Epoxy (filler)
Minimum Thickness (mils of) 1st Coat	0.5	0.5	0.5	0.5	0.5		0.5	0.5	0.5	0.5	0.5
1st Coat ^(c)	Epoxy (sealer)	Epoxy (sealer)	Epoxy (sealer)	Epoxy (sealer)	Epoxy (sealer)		Epoxy (sealer)	Epoxy (sealer)	Epoxy (sealer)	Epoxy (sealer)	Epoxy (sealer)
Tolerance Surface Area(%)	±10	± 10	±10	±10	±10		±10	±10	∓10	±10	±10
Estimated Surface Area (ft²) ^(b)	216(9)	1610(4)	625(4)	210(4)	940(4)		580(4)	1089(4)	229(4)	678(4)	315(4)
Thickness (in.)	48.0	24.0	33.0	36.0	0.09		18.0	36.0	36.0	176.0	36.0
<u>Material^(a)</u>	Concrete	Concrete	Concrete	Concrete	Concrete		Concrete	Concrete	Concrete	Concrete	Concrete
Item	No. 3 ceiling slab Not used	Operating deck slab - 220 ft 0 in. 2-ft 0-in. slab, 180° to 360°	2-ft 9-in. slab at 90°	3-ft 0-in. slab at 90° and 260°	5-ft 0-in. slab at 0° and 180°	Miscellaneous Walls	1-ft 6-in. wall at 90°	3-ft 0-in. wall	3-ft 0-in. wall - below R.F. canal	Mass concrete	Walls under R.F. canal

TABLE 6.1.2-2 (SHEET 5 OF 10)

Minimum Thickness (mils) of 4th Coat	3.0	3.0	3.0	3.0	3.0				3.0			
4th Coat	Epoxy (finish)	Epoxy (finish)	Epoxy (finish)	Epoxy (finish)	Epoxy (finish)				Epoxy (finish)			
Minimum Thickness (mils) of 3rd Coat	10.0	10.0	10.0	10.0	10.0				15.0			
3rd Coat	Epoxy (surface r)	Epoxy (surface r)	Epoxy (surface r)	Epoxy (surface r)	Epoxy (surface r)				Epoxy (surface r)			
Minimum Thickness (mils) of 2nd Coat 3	As needed to fill holes	As needed to fill holes	As needed to fill holes	As needed to fill holes	As needed to fill holes				As needed to fill holes			
2nd Coat	Epoxy (filler)	Epoxy (filler)	Epoxy (filler)	Epoxy (filler)	Epoxy (filler)				Epoxy (filler)			
Minimum Thickness (mils of) 1st Coat	O S	0.5	0.5	0.5	0.5		0.5	0.5	0.5			2.5
1st Coat ^(c)	Epoxy (sealer)	Epoxy (sealer)	Epoxy (sealer)	Epoxy (sealer)	Epoxy (sealer)		Epoxy (sealer)	Epoxy (sealer)	Epoxy (sealer)		Galvanized	Inorganic zinc
Tolerance Surface <u>Area(%)</u>	±10	±10	±10	±10	+10		±10	±10	±10		+15	±15
Estimated Surface Area (ff ²) ^(b)	1307(4)	2206(9)	137(9)	1400(9)	578(9)		1391(9)	63(9)	200(4)		86,630(4)	1328(4)
Thickness (in.)	30.0	12.0	18.0	12.0	18.0		12.0	18.0	24.0		0.19	0.22
<u>Material^(a)</u>	Concrete	Concrete	Concrete	Concrete	Concrete		Concrete	Concrete	Concrete		SS	S
<u>Item</u>	Instrument walls Stairs and Elevator Shaft Walls	Walls	Celling Stair No. 2	1-ft 0-in.	1-ft 6-in. wall and ceiling	Elevator	Walls	Celling	Miscellaneous pads	Structural Steel	Platform grating	Ladders and stairways

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TABLE 6.1.2-2 (SHEET 6 OF 10)

Minimum Thickness (mils) of 4th Coat																		
4th Coat																		
Minimum Thickness (mils) of 3rd Coat																		
3rd Coat																		
Minimum Thickness (mils) of 2nd Coat																		
2nd Coat																		
Minimum Thickness (mils of) 1st Coat		2.5	ic 2.5		2.5		2.5	2.5			2.5		2.5	2.5	2.5		2.5	2.5
1st Coat ^(c)		Inorganic zinc	Inorganic zinc		Inorganic zinc	Galvanized	Inorganic zinc	Inorganic zinc		Galvanized	Inorganic	7 7	Inorganic	Inorganic	Inorganic	ZIIIC	Inorganic	Inorganic
Tolerance Surface Area(%)		±20	±20		±10	±10	±10	±10		+100	+100	07-	±10	±10	±10		±10	+15
Estimated Surface Area (ft²) ^(b)		340(4)	15,670(4)		914(4)	5060(4)	2588(4)	840(4)		3250(4)	30,000(4)		2714(4)	980(4)	14,950(4)		14,625(4)	6194(4)
Thickness (in.)		7:	3.0		1.0	0.19	0.5	2.5		0.19	0.5		1.25	1.5	1.25		1.2	0.38
<u>Material^(a)</u>		S	S		S	S	CS	SS	nerator	CS	CS		SS	SS	SS	cture	S	CS
ltem	Pipe whip restraints	Light and medium capacity	Heavy and extra heavy capacity	Pressurizer support steel	Pressurizer supports	Grating	Platform beams	Crossover leg support	Platform at steam generator	Grating	Support beams	Polar crane runway	Brackets - 37	Rail	Girders - 37	Internal platform structure	Columns	Beams

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TABLE 6.1.2-2 (SHEET 7 OF 10)

Minimum Thickness (mils) of 4th Coat																			
4th Coat																			
Minimum Thickness (mils) of 3rd Coat																			
3rd Coat																			
Minimum Thickness (mils) of 2nd Coat																			
2nd Coat																			
Minimum Thickness (mils of) 1st Coat	2.5	2.5	2.5	2.5	2.5							2.5	2.5	2 2 2 2 5 5 5 5					
1st Coat ^(c)	zinc Inorganic	ZINC Inorganic	Zinc Inorganic	zinc Inorganic zino	zinc Inorganic zinc	Galvanized	Galvanized	Galvanized	Galvanized	Galvanized	NC	Inorganic zinc	Inorganic zinc	Inorganic zinc Inorganic zinc Inorganic zinc	NC	NC	N N		NC
Tolerance Surface Area(%)	±15	±15	±15	±15	+1 52	ı	±20	±20	ı	ı	ı	±20	+ 2	H H H	+ 2	+ 2	+ 2	+ 2	H 1+
Estimated Surface Area (ft²) ^(b)	13,096(4)	37,991(4)	13,012(4)	12,741(4)	3141(4)	20,697	7900(4)	3500(4)	12,779	6141	12,000	2000(4)	51,787	13,750 2970 386	182	205	194	53	387
Thickness (in.)	0.5	0.62	0.75	1.0	د .	0.0613	0.38	0.19	0.15	0.105	0.1	0.38	0.25	1.25 0.5 0.5	0.375	0.312	1.312	1.125	0.688
Material ^(a)	cs	CS	CS	CS	CS	CS	CS	CS	CS	CS	SS	SS	CS	S S S S	SS	SS	SS	SS	SS
<u>Item</u>						Cable trays	Cable tray support		Conduit	Conduit Boxes	HVAC ducting	HVAC ducting bracing and hangers	Pipe supports (steel members)	Pipe racks Snubbers Spring hangers Uninsulated piping	24 in.	14 in.	12 in.	12 in.	12 in.

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TABLE 6.1.2-2 (SHEET 8 OF 10)

Minimum Thickness (mils) of 4th Coat																				
4th Coat																				
Minimum Thickness (mils) of 3rd Coat																				
3rd Coat																				
Minimum Thickness (mils) of 2nd Coat																				
2nd Coat																				
Minimum Thickness (mils of) 1st Coat		2.5	2.5			2.5	2.5				2.5	2.5				2.5	2.5	2.5	2.5	
1st Coat ^(c)	N	Inorganic zinc NC	Inorganic zinc NC	NC	NC	Inorganic zinc Inorganic zinc	Inorganic	NC O	NC	NC	Inorganic	Zinc Inorganic	NC OC	NC	NC	Inorganic	zinc Inorganic zinc	Inorganic zinc	Inorganic zinc	OZ
,,	± 5 NC	± 5 Inorganic zinc ± 5 NC	±5 Inorganic zinc ±5 NC	+ 5 NC	± 5	± 5 Inorganic zinc ± 5 Inorganic zinc	± 5 Inorganic	± 5 NC	± 5 NC	± 5	± 5 Inorganic	±5 Inorganic	± 5 NC	± 5	± 5	±5 Inorganic	± 5 Inorganic zinc	± 5 Inorganic zinc	± 5 Inorganic zinc	+ 2
1st Coat ^(c)	22	2 2	വവ	2	2		2	2	2	5	2	2	5	5	5	2	ري د	2	2	
Tolerance Surface Area(%) 1st Coat ^(c)	+ +	+ + 5	H H	+ 2	+ 2	H H	ις Η	+ 5	+ 5	+ 5	15	+1	+ 22	H 1	± 5	+ 2	H 22	£ +1	C) H	+ 2
Estimated Surface Tolerance Area Surface (ft²)(b) Area(%) 1st Coat(c)	367 ± 5	206 ±5 1509 ±5	832 ± 5 4769 ± 5	286 ± 5	5752 ± 5	1139 ± 5 2530 ± 5	9 # 2	2806 ± 5	1289 ±5	210 ±5	119 ±5	203 ±5	58 ± 5	1146 ±5	388 ±5	451 ±5	267 ±5	71 ±5	12 ± 5	733 ± 5

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TABLE 6.1.2-2 (SHEET 9 OF 10)

Minimum Thickness (mils) of 4th Coat																						
4th Coat																						
Minimum Thickness (mils) of 3rd Coat																						
3rd Coat																						
Minimum Thickness (mils) of 2nd Coat																						
2nd Coat																						
Minimum Thickness (mils of) 1st Coat				2.5	2.5				2.5	2.5	2.5							2.5				
- F = F																						
1 (()	NO	NC	NC	Inorganic	Zinc Inorganic Zinc	NC	NC	NC	Inorganic	Inorganic	Zinc Inorganic	NC	NC	NC	NC	NC		Inorganic	NC	NC	NC	NC
, 0.1	+ 5	± 5	± 5	± 5 Inorganic	± 5 Inorganic zinc	+ 5 NC	+ 5	± 5 NC	± 5 Inorganic	± 5 Inorganic	± 5 Inorganic	± 5 NC	± 5 NC	± 5 NC	± 5	± 5 NC	± 5	± 5 Inorganic	± 5 NC	+ 5	+ 5	± 5 NC
1st Coat ^(c)	22	2	2	2	rC	5	5	2	2	2	22	2	2	2	2	5		5	2	2	2	2
Tolerance Surface (Area(%) 1st Coat ^(c)	± 5	+ 2	H H	ις H	4	+ 5	+ 5	+ 2	+ 2	++	H 2	+ 5	+ 2	+ 2	+ 5	ις Η	+1	5	ις +1	1+ 22	+ 2	H 5
Estimated Surface Tolerance Area Surface (ft²)(b) Area(%) 1st Coat(c)	512 ± 5	25 ± 5	21 ±5	36 ± 5	421 ± 5	48 ±5	463 ± 5	627 ± 5	35 ± 5	783 ±5	14 ±5	443 ±5	72 ±5	190 ± 5	303 ±5	407 ±5	∓ 926	1 10	140 ± 5	15 + 5	278 ±5	123 ± 5

TABLE 6.1.2-2 (SHEET 10 OF 10)

<u>ltem</u>	<u>Material^(a)</u>	Thickness (in.)	Estimated Surface Area (ft²) ^(b)	Tolerance Surface Area(%)	1st Coat ^(c)	Minimum Thickness (mils of) 1st Coat	2nd Coat	Minimum Thickness (mils) of 2nd Coat	3rd Coat	Minimum Thickness (mils) of 3rd Coat	4th Coat	Minimum Thickness (mils) of 4th Coat
3/8 in.	SS	0.065	38	+ 2	NC							
Containment	cs	0.25	20,500	+10	Inorganic	2.5	Epoxy	3.0				
Containment	CS	0.25	4400	+10	Inorganic	2.5	Epoxy	3.0				
duxiliary coolers Cavity cooling coils	SO	0.25	2200	+10	Inorganic	2.5	Epoxy	3.0				
ESF fans	CS	0.25	200	+10	Inorganic zinc	2.5	Epoxy	3.0				

a. CS - carbon steel SS - stainless steel

Interior surface (liner plate) backed by concrete. Only one side directly exposed to containment environment. Surface area of exposed side given. b. (5) (5) (4)

Interior surface (liner plate) backed by concrete and covered with filler slab (concrete). Not directly exposed to containment environment. Surface area of one side given.

Direct boundary between containment interior and exterior environment. Interior surface area given.

Specified member (beams, walls, slabs, etc.) completely exposed (all sides) to the containment environment. Total exposed surface area given (e.g., for walls both sides are given;

for wide flange shapes both sides of flanges and web are given).

(5) Exterior surface. One side directly exposed to exterior environment. Surface area of exposed side given.

Interior surface covered with steel liner plate. Not directly exposed to containment environment. Interior surface area given.

Interior surface. One side directly exposed to containment environment with basemat steel liner plate on opposite side. Exposed surface area given.

Interior surface. One side directly exposed to containment environment with steel liner on opposite side (liner in turn exposed to containment environment). Surface area of directly exposed side given. (9)(2)(8)

Outer surface directly exposed to general containment environment with inner area almost entirely self-enclosed. Directly exposed outer surface area given. 6

c. NC denotes no coating.

d. This epoxy is applied as a wainscot which covers containment interior walls from elevations 171 ft 9 in. to 181 ft 9 in. and elevations 220 ft to 228 ft. The remainder of concrete walls are coated with 0.5 mils of epoxy only.

e. Although actual areas for individual categories may differ slightly from the values shown, the total surface areas are considered to be within the indicated tolerances.

TABLE 6.1.2-3

PROTECTIVE COATINGS ON WESTINGHOUSE-SUPPLIED EQUIPMENT INSIDE CONTAINMENT

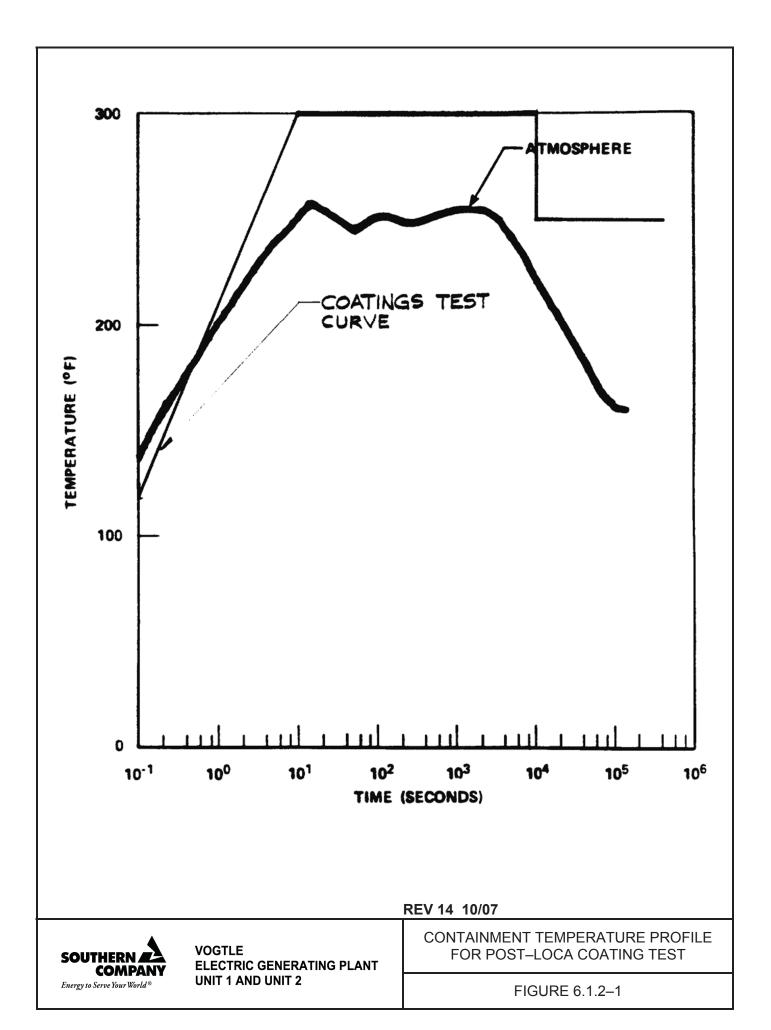
Component	Estimated Coated Surface Area (ft²)	
Reactor coolant system component supports	11,000 ^(a)	
Reactor coolant pump assemblies	5200	
Accumulator tanks	5400	
Refueling machine	3925	
Other refueling equipment	2125	
Remaining equipment (such as valves, auxiliary tanks and heat exchanger supports, transmitters, alarm horns, small instruments, etc.)	<1300	

a. Primer only.

TABLE 6.1.2-4

OTHER ORGANIC MATERIALS

<u>Item</u>	<u>Material</u>	Quantity Enclosed (lb)	Quantity Exposed (lb)
Cable insulation	Ethylene propylene rubber and chlorosul-phonated polyethylene	16,100	24,800
Heat shrink tubing	Raychem WCSF-N	650	-
Lug insulation	AMP special indus- tries type PVF	200	-
Cable ties	Thomas and Bets Tefzel	100	500
Terminal blocks	Diallyl-phthalate long glass fiber fill	400	-



6.2 CONTAINMENT SYSTEMS

The containment systems include the containment, the containment heat removal systems, the containment isolation system, and the containment combustible gas control system.

The design basis accident (DBA) is defined as the most severe of a spectrum of hypothetical loss-of-coolant accidents (LOCA) and high-energy line breaks within the containment. The ability of the containment systems to mitigate the consequences of a DBA depends upon the high reliability of these systems. This section provides the design criteria and evaluations to demonstrate that these systems function within the specified limits throughout the unit operating lifetime^a.

6.2.1 CONTAINMENT FUNCTIONAL DESIGN

6.2.1.1 <u>Containment Structure</u>

6.2.1.1.1 Design Bases

The containment system-is designed such that for all break sizes, up to and including the double-ended severance of a reactor coolant pipe or secondary system pipe, the containment peak pressure is below the design-pressure with an adequate margin, as presented in table 6.2.1-1.

The analyses presented in this section are based on assumptions that are conservative with respect to the containment and its heat removal systems; i.e., minimum heat removal, maximum containment pressure.

This capability is maintained by the containment system even assuming the worst single active failure affecting the operation of the emergency core cooling system (ECCS), containment spray system, and reactor containment fan coolers during the injection phase and the worst active or passive single failure during the recirculation phase. For primary system breaks, loss-of offsite power (LOSP) is assumed. For secondary system breaks, LOSP is not assumed, since this would reduce releases to the containment.

Paragraphs 6.2.1.3 and 6.2.1.4 present the mass and energy releases used in the design evaluation.

An evaluation was conducted to determine the impact of the uprating/ T_{hot} reduction program conditions on the short term LOCA mass and energy releases used for the subcompartment analyses. The evaluation is discussed in paragraph 6.2.1.2.3.2.1.

6.2.1.1.2 Design Features

Containment design features are further described in the following:

^a The operating licenses for both VEGP units have been renewed and the original licensed operating terms have been extended by 20 years. In accordance with 10 CFR Part 54, appropriate aging management programs and activities have been initiated to manage the detrimental effects of aging to maintain functionality during the period of extended operation (see chapter 19).

- A. A secondary shield wall constructed of a minimum of 3-ft-thick reinforced concrete, extends from the basement filler slab (171 ft 9 in.) to the operating deck (220 ft). The shield wall encloses the reactor coolant pumps, reactor vessel and its primary shield wall, steam generators up to 238 ft, the pressurizer up to 268 ft, and the refueling cavity. The secondary shield wall supports the operating deck, which together with the shield wall, prevents the containment liner from being impacted by potential internal missiles and the effects of-pipe whip. Refer to sections 3.5 and 3.6 for more detailed discussion.
- B. The reactor containment is designed and constructed in accordance with the codes and standards discussed in subsections 3.8.1 and 3.8.3.
- C. The VEGP design does not use a pressure-suppression-type containment.
- D. Inasmuch as the reactor containment fan coolers are utilized during normal operation, inadvertent changes to the accident mode produce no significant effect upon containment internal pressure.
- E. The only location inside the containment where water may be trapped and prevented from returning to the containment emergency sumps is the reactor cavity. Water can enter the cavity by flowing down through the ventilation openings surrounding the reactor cavity seal ring. Approximately 113,200 gal of water could collect in the reactor cavity. This would cause the static head for the residual heat removal (RHR) and spray pumps to decrease by about 1.31 ft. This decrease would not impair the operation of these pumps. Most of the water entering the refueling canal is returned to the containment emergency sumps via two, normally open, 12-in. drain lines.
- F. The containment and subcompartment atmospheres are maintained during normal operation within prescribed pressure, temperature, and humidity limits by means of the nuclear service cooling water (NSCW) system which delivers 90°F water to the cooling coils within each containment fan cooler. Containment ventilation systems, such as the control rod drive mechanism booster fans and cooling fans, are used during normal operation and require no periodic testing to ensure functional capability.

6.2.1.1.3 Design Evaluation

The short-term pressure subcompartment analysis considers a LOSP. Consideration of single active failures is of no consequence, since none of the safety equipment functions during the initial seconds of the post-accident transient. The maximum calculated differential pressure in the steam generator compartment is 23.50 psid resulting from a 436 in.² in area guillotine rupture in the steam generator outlet nozzle. The maximum calculated differential pressure in the upper pressurizer cubical is 5.84 psid resulting from a spray line double-ended break. The maximum calculated differential pressure in the lower pressurizer cubicle is 20.7 psid from a surge line double-ended break. The containment subcompartment differential pressure analysis is described in detail in paragraph 6.2.1.2.

An evaluation was conducted to determine the impact of the uprating/T_{hot} reduction program conditions on the short term LOCA mass and energy releases used for the subcompartment analyses. The evaluation is discussed in paragraph 6.2.1.2.3.2.1.

The results of the pressure transient analysis of the containment for the LOCA are shown in figures 6.2.1-1 through 6.2.1-3. Containment temperature curves are presented in figures 6.2.1-

4 through 6.2.1-6. The cases examined in this analysis determine the effects of the full range of large reactor coolant break sizes up to and including a double-ended rupture. Cases illustrating the sensitivity to break location are also shown. All of these cases show that the containment pressure will remain below design pressure with margin. After the peak pressure is attained, the operation of the safeguards system reduces the containment pressure. At the end of the first day following the accident, the containment pressure has been reduced to a low value. The peak pressures are shown in table 6.2.1-1.

Calculation of containment pressure and temperature transients is accomplished by use of the digital computer code COCO.⁽¹⁾ The COCO code has been used and found acceptable to calculate containment pressure transients for the H. B. Robinson (docket number 50-261) and Zion (docket number 50-295) plants. Transient phenomena within the reactor coolant system (RCS) affect containment conditions by means of convective mass and energy transport through the pipe break.

For analytical rigor and convenience, the containment air-steam-water mixture is separated into two systems. The first system consists of the air-steam phase; the second is the water phase. Sufficient relationships to describe the transients are provided by the equations of conservation of mass and energy as applied to each system, together with appropriate boundary conditions. Since thermodynamic equations of state and conditions may vary during the transient, the equations have been derived for all possible cases of superheated or saturated steam and subcooled or saturated water. Switching between states is handled automatically by the code. The following are the major assumptions made in the analysis.

- A. Discharge mass and energy flowrates through the RCS break are established from the analysis in paragraph 6.2.1.3.
- B. For the steam line break analysis and the blowdown portion of the LOCA analysis, the discharge flow separates into steam and water phases at the breakpoint. The saturated water phase is at the total containment pressure, while the steam phase is at the partial pressure of the steam in the containment. For the post-blowdown portion of the LOCA analysis, steam and water releases are input separately.
- C. Homogeneous mixing is assumed. The steam-air mixture and the water phase each have uniform properties. More specifically, thermal equilibrium between the air and steam is assumed. This does not imply thermal equilibrium between the steam-air mixture and water phase.
- D. Air is taken as an ideal gas, while compressed water and steam tables are employed for water and steam thermodynamic properties.
- E. For large steam line ruptures the saturation temperature at the partial pressure of the steam is used for heat transfer to the heat sinks and the fan coolers.

Paragraphs 6.2.1.3 and 6.2.1.4 present the mass and energy releases used for the analysis.

6.2.1.1.3.1 <u>Initial Conditions</u>. An analysis of containment response to the rupture of the RCS must start with knowledge of the initial conditions in the containment. The pressure, temperature, and humidity of the containment atmosphere prior to the postulated accident are specified in the analysis.

Also, values for the temperature of the NSCW and refueling water storage tank solution are assumed, along with the initial water inventory of the refueling water storage tank. All of these values are chosen conservatively, as shown in table 6.2.1-2.

Assumptions for containment analysis are shown in table 6.2.1-3. The assumed spray flowrate is based on one of two trains operating.

6.2.1.1.3.2 <u>Heat Removal</u>. The significant heat removal source during the early portion of the transient is structural heat removal. Provision is made in the containment pressure transient analysis for heat transfer through, and heat storage in, both interior and exterior walls. Every wall is divided into a large number of nodes. For each node, a conservation of energy equation expressed in finite-difference form accounts for transient conduction into and out of the node and temperature rise of the node. Tables 6.2.1-4 and 6.2.1-5 are summaries of the containment structural heat sinks used in the analysis.

The heat transfer coefficient to the containment structure is calculated by the code based primarily on the work of Tagami. From this work, it was determined that the value of the heat transfer coefficient increases parabolically to peak value at the end of blowdown for LOCA and increases parabolically to peak at the time of steam line isolation. The value then decreases exponentially to a stagnant heat transfer coefficient which is a function of steam to air weight ratio.

Tagami presents a plot of the maximum value of h as a function of "coolant energy transfer speed," defined as follows:

total coolant energy transferred into containment (containment volume) (time interval to peak pressure

From this the maximum of h steel is calculated:

$$h_{\text{max}} = 75 \left(\frac{E}{t_{\rho} V} \right)^{0.6} \tag{1}$$

where:

 h_{max} = Maximum value of h (Btu/h-ft²-°F).

t_ρ = Time from start of accident to end of blowdown for LOCA and steam line isolation for secondary breaks (s).

V = Containment volume (ft³).

E = Coolant energy--discharge (Btu).

The parabolic increase to the peak value is given by:

$$h_s = h_{\text{max}} \left(\frac{t}{t_{\rho}} \right)^{0.5} \quad (0 \le t \le t_{\rho})$$
 (2)

where:

 h_s = Heat transfer coefficient for steel (Btu/h-ft²- $^{\circ}$ F).

t = Time from start of accident (s).

For concrete, the heat transfer coefficient is taken as 40% of the value calculated for steel.

The exponential decrease of the heat transfer coefficient is given by:

$$h_s = h_{stag} + (h_{max} - h_{stag})_{e^{-0.05(t-t_\rho)}} (t > t_o)$$

where:

 $h_{stag} = 2 + 50 X \quad (0 \le X \le 1.4).$

 h_{stag} = h for stagnant conditions (Btu/h-ft² °F).

x = Steam to air weight ratio in containment.

For a large break the safety features are quickly brought into operation. Because of the brief period of time required to depressurize the RCS, the safeguards are not a major influence on the blowdown peak pressure; however, they reduce the containment pressure after the blowdown and maintain a low long-term pressure. Also, although the containment structure is not as effective a heat sink as during the RCS blowdown, it still contributes significantly as a form of heat removal during the long-term cooling period.

During the injection phase of post-accident operation, the ECCS pumps water from the refueling water storage tank into the reactor vessel. Since this water enters the vessel at refueling water storage tank temperature, which is less than the temperature of the water in the vessel, it can absorb heat from the core until saturation temperature is reached. During the recirculation phase of operation, water is taken from the containment sump and cooled in the residual heat exchanger.

The cooled water is then pumped back to the reactor vessel to absorb more decay heat. The heat is removed from the residual heat exchanger by component cooling water.

Another containment heat removal system is the containment spray. During the injection phase of operation, the containment spray pumps draw water from the refueling water storage tank and spray it into the containment through nozzles mounted high above the operating deck. As the spray droplets fall, they absorb heat from the containment atmosphere. Since the water comes from the refueling water storage tank, the entire heat capacity of the spray from the refueling water storage tank temperature to the temperature of the containment atmosphere is available for energy absorption. During the recirculation phase of post-accident operation, water is drawn from the sump and sprayed into the containment atmosphere.

When a spray drop enters the hot, saturated, steam-air containment environment following a LOCA, the vapor pressure of the water at its surface is much less than the partial pressure of the steam in the atmosphere. Hence, there will be diffusion of steam to the drop surface and condensation on the drop. This mass flow will carry energy to the drop. Simultaneously, the temperature difference between the atmosphere and the drop will cause the drop temperature and vapor pressure to rise. The vapor pressure of the drop will eventually become equal to the partial pressure of the steam, and the condensation will cease. The temperature of the drop will essentially equal the temperature of the steam-air mixture.

The equations describing the temperature rise of a falling drop are as follows:

$$\frac{d}{dt}(Mu) = mh_g + q \tag{3}$$

$$\frac{d}{dt}(Mu) = m \tag{4}$$

where:

$$q = h_c A (T_s - T).$$

$$m = k_a A (P_s - P_v).$$

The coefficients of heat transfer (h_c) and mass transfer (k_g) are calculated from the Nusselt number for heat transfer, <u>Nu</u>' and the Nusselt number for mass transfer, <u>Nu</u>'.

Both Nu and Nu may be calculated from the equations of Ranz and Marshall⁽³⁾.

Nu
$$2 + 0.6$$
 (Re) $\frac{1}{2}$ (Pr) $\frac{1}{3}$ (5)

$$Nu'$$
 2 + 0.6 (Re) $\frac{1}{2}$ (Sc) $\frac{1}{3}$ (6)

Thus, equations 3 and 4 can be integrated numerically to find the internal energy and mass of the drop as a function of time as it falls through the atmosphere. Analysis shows that the temperature of the (mass) mean drop produced by the spray nozzles rises to a value within 99% of the bulk containment temperature in less than 2 s.

Drops of this size will reach temperature equilibrium with the steam-air containment atmosphere after falling through less than half the available spray fall height.

Detailed calculations of the heatup of spray drops in post-accident containment atmospheres by Parsly⁽⁴⁾ show that drops of all sizes encountered in the containment spray reach equilibrium in a fraction of their residence time in a typical pressurized water reactor containment.

These results confirm the assumption that the containment spray will be 100% effective in removing heat from the atmosphere.

Nomenclature relevant to the above discussion is listed below:

A = Area.

h_c = Coefficient of heat transfer.

k_q = Coefficient of mass transfer.

 h_{α} = Steam enthalphy.

M = Droplet mass.

m = Diffusion rate.

Nu = Nusselt number for heat transfer.

Nu' = Nusselt number for mass transfer.

P_s = Steam partial pressure.

 P_v = Droplet vapor pressure.

Pr = Prandtl number.

q = Heat flowrate.

Re = Reynolds number

Sc = Schmidt number.

 T_s = Droplet temperature.

T = Steam temperature.

t = Time.

u = Internal energy.

The reactor containment fan coolers are a final means of heat removal. The main aspects of a fan cooler from the heat removal standpoint are the fan and the banks of cooling coils. The fans draw the dense atmosphere through banks of finned cooling coils and mix the cooled steam-air mixture with the rest of the containment atmosphere. The coils are kept at a low temperature by

a constant flow of cooling water. Since this system does not use water from the refueling water storage tank, the mode of operation remains the same both before and after the spray system and ECCS change to the recirculation mode. Fan cooler heat removal performance is shown in figure 6.2.1-7.

6.2.1.1.3.3 <u>Inadvertent Spray Actuation</u>. In the event of inadvertent spray, the containment will depressurize until the air temperature is approximately equal to the spray temperature or the operator takes action to terminate the spray.

The COCO computer code was used to calculate the minimum pressure inside the containment. The following assumptions were made:

- A. The spray flowrate is 6748 gal/min and is at 40°F.
- B. The containment is initially at 120°F, 14.093 psi, and has 100% humidity.
- C. Operator action to stop the spray occurs at 10 min.

The containment pressure reduces to 11.77 psia at 10 min into the transient. Thus, the peak differential pressure is 2.93 psi across the containment shell.

- 6.2.1.1.3.4 <u>Inadvertent Purge Actuation</u>. In the event of inadvertent containment purge exhaust actuation, the maximum differential pressure that can be drawn on the containment is 0.9 psi. The following assumptions were made:
 - A. The preaccess purge fan is assumed to be actuated because it has a larger capacity than the minipurge fan (subsection 9.4.6).
 - B. The preaccess purge fan is operating during mode 5 or 6. During modes 1 through 4, the preaccess purge penetrations are sealed closed.
 - C. There is no air supply or air leakage into the containment.
 - D. The containment is initially at 14.7 psia.

Consequently, the maximum differential pressure at the onset of a LOCA would be less than 0.9 psi.

In the VEGP minimum containment pressure analysis, an initial pressure of 14.7 psia (0.0 psig) has been assumed. This assumption is valid because it represents the containment pressure condition consistent with normal, full power operation of the plant. As a rule, parameters input to a 10 CFR 50.46 ECCS performance analysis correspond with expected, nominal plant conditions at 100% power operation. The large degree of conservatism present in 10 CFR 50.46 evaluation model analyses (as compared to historical best estimate results) was intended and does in fact cover uncertainties in the prevailing plant operating parameters at the time a postulated LOCA occurs. Therefore, utilization of atmospheric pressure, which is representative of the normal, full power containment condition, is appropriate for Vogtle and is consistent with the LOCA evaluation model philosophy.

Furthermore, at 100% power, the reactor coolant system is a significant heat source which heats and pressurizes air present in the containment. In the event that such pressurization occurs at Vogtle, containment purging will be conducted. In paragraph 6.2.1.5.8, a specific evaluation documents the penalty in calculated ECCS performance which is incurred when a LOCA occurs coincident with containment purging. This penalty is applied to the VEGP limiting case result. Since the very operation of the containment purge system is predicated upon a

high prevailing containment pressure, the consideration of containment purging being conducted at the time of a large break LOCA together with the assumption of 0 psig containment initial pressure accommodates any need for conservatism in the Vogtle ECCS performance analysis containment pressure computation.

Technical Specification limits on containment internal pressure cover operation through Mode 4. As discussed above, use of 0 psig is appropriate at the 100% power operation condition at which the VEGP ECCS performance analysis is conducted.

The 0 psig value of initial containment pressure is traditionally applied in 10 CFR 50.46 ECCS performance analyses.

In paragraph 6.2.1.5.8 is included an evaluation of purge operation during a double-ended cold leg guillotine break.

6.2.1.1.3.5 <u>Accident Chronology</u>. The accident chronology for the limiting LOCA is given in table 6.2.1-6.

6.2.1.2 Containment Subcompartments

6.2.1.2.1 Design Bases

Subcompartments within containment (principally, the reactor cavity, the steam generator compartments, and the pressurizer compartment) are designed to withstand the transient differential pressures and jet impingement forces of a postulated pipe break. Venting of these chambers is employed to keep the differential pressures within structural limits. In addition, restraints on the coolant pipes, reactor vessel, steam generators, etc., are designed to limit pipe whip effects and forces transmitted through component supports to ensure the integrity of subcompartments and the containment structure.

- 6.2.1.2.1.1 <u>Summary of Subcompartment Pipe Break Analyses</u>. The postulated breaks in the high-energy lines that are analyzed to determine the maximum differential pressure across the subcompartment walls are tabulated in table 6.2.1-8. The characteristics of the main coolant pipe ruptures are determined in accordance with the methods and criteria of subsection 3.6.2.
- 6.2.1.2.1.2 <u>Reactor Vessel Cavity</u>. The pipe restraints provided for the RCS limit the size of a break in the cavity region to 144 in.² The blowdown of coolant from a cold leg results in the highest pressure loadings on the reactor cavity walls and the reactor vessel. Accordingly, the analysis of the reactor cavity break assumes that the break occurs in a cold leg pipe.

A multi-compartment nodal model of the reactor cavity around the reactor vessel, reactor coolant piping nozzles, and connected volumes was developed in accordance with the guidelines presented in NUREG-0609. The subcompartment boundaries are set at planes of natural obstructions to flow.

The nodal network and the break mass/energy release rate (supplied by the nuclear steam supply system vendor) are input to the computer program COPDA, NE-699-D2, which performs

a stepwise calculation of the subcompartment pressures and temperatures as a function of time following the line break. Subcompartment pressures are used to calculate pressure differentials across the cavity structural members and force loadings on the reactor vessel.

In the development of the reactor cavity model, the following assumptions are made regarding the reactor vessel thermal insulation.

- A. For the purpose of node volume calculation, all thermal insulation (uniform 4 in. thick) remains in place.
- B. The upper reactor seal ring is completely plugged.
- C. Fifty% of the annular space between the vessel and cavity wall is plugged along the plane perpendicular to the break location.
- D. Fifty% of the remaining flow area in the instrument tunnel, after considering the steel of the first bottom mounted instrument support is plugged.
- E. Access ports to annular inspection cavity are closed.
- F. The nozzle penetrations into the cavity on the unaffected loops are completely plugged. The nozzle penetration into the cavity on the affected loop is open.
- G. The nozzle penetrations to the steam generator compartments are open along the unaffected loops and plugged along the affected loop.

Figure 6.2.1-15 shows the resultant pressures in the central volumes shown in the nodal model in figure 6.2.1-16. The flow model is given in tables 6.2.1-9 and 6.2.1-10.

A summary of the forces and moments acting on the reactor vessel is included in table 6.2.1-73. Nodes 1 through 48 (figure 6.2.1-16) are those that contribute to F_x , F_g , F_{total} , M_X , and M. The magnitude of these loads is given at the axial centroids of the node groups. Uplift forces and the moment about the vessel axis are provided in table 6.2.1-74.

- 6.2.1.2.1.3 <u>Steam Generator Compartment</u>. For the analysis of the pressure transient in the steam generator compartment following a line break, the flow model, including control volumes, intercompartment flowpaths, and corresponding flow coefficients) is illustrated in figures 6.2.1-16A and 17 and tables 6.2.1-11, 11A through 6.2.1-22. The piping restraints on the reactor coolant loop restrict the motion of the piping following a pipe rupture in the primary loop. This restriction limits the size of the effective blowdown areas for postulated breaks in the steam generator compartments to an area less than, or equal to, a single-ended flow area of the pipe. The calculated steam generator compartment pressure response is shown in figures 6.2.1-17A through 6.2.1-23.
- 6.2.1.2.1.4 <u>Pressurizer Compartment</u>. For the analysis of the pressure transient following a pressurizer surge line or a spray (or relief) line break, the flow model of control volumes, intercompartment flowpaths, and corresponding flow coefficients are illustrated in figure 6.2.1-24 and tables 6.2.1-23 through 6.2.1-25. The calculated compartment pressure response is shown on figures 6.2.1-25 and 6.2.1-25a for the surge line break and the spray line break at the pressurizer nozzle, respectively.

6.2.1.2.2 Design Features

The effects of high-energy line breaks in the reactor cavity, steam generator compartments, and pressurizer compartment were analyzed to establish criteria for the structural design of the compartment walls. Plans and sections of these compartments are shown in drawings 1X2D01A001 and 1X2D01J015. The models of these compartments as used in the analyses are shown in figures 6.2.1-16, 6.2.1-16A, 17, and 6.2.1-24. The node volumes, flowpaths, flowpath areas, and flow coefficients for each of the compartments are tabulated in tables 6.2.1-9 through 6.2.1-25.

6.2.1.2.3 Design Evaluations

- 6.2.1.2.3.1 <u>Analytical Model for Subcompartment Analyses</u>. A digital computer program was used to perform the short-term compartment pressure transient analysis. This program is capable of handling up to 100 control volumes with a maximum of five flowpaths out of any compartment. The calculational methods used in this computer program are described in detail in BN-TOP-4, Rev. 1.
- 6.2.1.2.3.2 <u>Analytical Model for Short Term Mass and Energy Releases</u>. The computer models, which were used to develop the mass and energy release transients for the subcompartment pressurization analysis, are described in reference 9. Tables 6.2.1-26 through 6.2.1-28a provide tabulations of the mass and energy release rates versus time for the spectrum of breaks which were analyzed.

For the reactor vessel cavity, the design basis break is a double-ended rupture of the reactor vessel inlet pipe which is restrained to 144 in.² (table 6.2.1-26).

For the steam generator compartment, the following breaks are considered:

- A. A double-ended reactor coolant pump outlet nozzle break restrained to 236 in.² (table 6.2.1-27, sheets 1 through 6).
- B. A double-ended generator inlet nozzle break restrained to 306 in.² (table 6.2.1-27, sheets 7 through 12).
- C. A double-ended break at the loop closure weld restrained to 336 in.² (table. 6.2.1-27, sheets 13 through 18).
- D. A double-ended reactor coolant pump inlet nozzle break restrained to 336 in.² (table 6.2.1-27, sheets 19 through 24).
- E. A double-ended steam generator outlet nozzle break restrained to 436 in.² (table 6.2.1-27, sheets 25 through 30).
- F. A steam-generator inlet elbow split break with a break area equal to 763 in.² (table 6.2.1-27, sheets 31 through 36).
- G. A double-ended main feedwater line break (table 6.2.1-26A).

For the pressurizer compartment, two break locations are considered:

- A. A double-ended surge line break (table 6.2.1-28).
- B. A double-ended spray line rupture (table 6.2.1-28a).

6.2.1.2.3.2.1 Effects of the Uprating/T_{hot} Reduction Programs on the LOCA Short Term Mass and Energy Releases. The short term mass and energy releases to containment, as a result of a LOCA, are described in paragraph 6.2.1.2.3.2. The blowdown mass and energy release rates are affected by the initial RCS temperature conditions. Since short term releases are linked directly to the critical mass flux, which increases with decreasing temperatures, the short term LOCA releases will increase due to the reductions in RCS coolant conditions associated with the uprating/T_{hot} reduction programs.

Releases currently described in paragraph 6.2.1.2.3.2 for the steam generator compartment have been evaluated for the uprating/T_{hot} reduction conditions. Based upon RCS temperature decreases of 15.0°F for the vessel outlet (from 618.2 to 603.2°F), 20.5°F for the vessel/core inlet (from 558.8 to 538.3°F), and 20.6°F for the steam generator outlet (from 558.6°F to 538.0°F), the releases in table 6.2.1-27, sheets 1 through 36 can increase by as much as 12%. Even though the current analysis includes 10% margin to account for uncertainties in the analytical model, noding effects, and initial reactor coolant conditions, the 10% margin may not be adequate to offset the 12% increase. However, per reference 14, VEGP has been granted an exemption from a portion of the requirements of GDC 4. This exemption eliminates the need to consider the dynamic effects associated with postulated pipe breaks in the primary loop. Therefore, the releases currently in table 6.2.1-27, sheets 1 through 36 remain bounding for smaller nozzles attached to the RCS, whenever uprating/T_{hot} reduction effects are considered.

For the reactor vessel cavity, based upon RCS temperature decreases of 28.0°F for the vessel inlet, which is the maximum difference between the current data used in the paragraph 6.2.1.2.3.2 analysis and the uprating/T_{hot} reduction data, the releases in table 6.2.1-26 can increase by as much as 6%. The current analysis includes 10% margin to account for uncertainties, which is adequate to offset the 6% increase. Additionally, per reference 15, the average break opening area for the inlet break is 58 in.². The current releases, which are based upon a 144 in.² break size, remain bounding considering uprating/T_{hot} reduction effects, whenever the actual break size based upon the structural evaluation is considered. Furthermore, per reference 14, VEGP has been granted an exemption from a portion of the requirements of GDC 4. This exemption eliminates the need to consider the dynamic effects associated with postulated pipe breaks in the primary loop.

For the pressurizer compartment, there are two break locations of importance; namely, the surge line and the spray line. For the surge line, based upon an RCS temperature decrease in the analysis of 20.7°F, the releases in table 6.2.1-28 can increase by as much as 16%. The current analysis, described in paragraph 6.2.1.2.3.2, includes 10% margin to account for uncertainties, which may not be adequate to offset this 16% increase. However, per references 16 and 17, VEGP has been granted an exemption from a portion of the requirements of GDC 4. This exemption eliminates the need to consider the dynamic effects associated with postulated rupture in the pressurizer surge line. For the spray line, the current releases are also expected to go up by 16%. However, upon closer inspection of the actual pipe sizes for the Vogtle spray line, and comparing to the current analysis break areas, it has been determined that the impact of the uprating/T_{hot} reduction is more than offset by the available margin in the break size, without the additional consideration of the 10% margin included in the current releases. Therefore, the releases in table 6.2.1-28a remain bounding for the spray line break.

In summary, an evaluation of the current short term LOCA mass and energy releases described in paragraph 6.2.1.2.3.2, utilized for the steam generator compartment, the reactor vessel cavity, and the pressurizer subcompartment, was conducted. It has been determined that the current short term LOCA mass and energy releases described in the previous section remain bounding for the uprating/ T_{hot} reduction conditions whenever the margin in the current calculations are considered and whenever leak-before-break is appropriately credited.

6.2.1.2.3.2.2 Evaluation for MUR Power Uprate - LOCA Short-Term Mass and Energy Releases. Short-term LOCA mass and energy release calculations are performed to support the reactor cavity and loop subcompartment pressurization analyses (which includes the steam generator compartment and pressurizer compartment).

The analysis inputs that may potentially change with the uprate are the initial reactor coolant system (RCS) fluid temperatures. Since this event lasts for approximately 3 seconds, the single effect of power is not significant. The approved methodology for the short-term LOCA mass and energy analysis is documented in reference 9. The critical flow calculation employs appropriately defined critical flow correlations applied for fluid conditions at the break location. Most short-term blowdown transients are characterized by a peak mass and energy release rate that occurs during a subcooled condition. The Zaloudek correlation, which models this condition, is currently used in the short-term LOCA mass and energy release analyses. The Zaloudek correlation was used to conservatively evaluate the impact of the changes in the RCS inlet and outlet temperatures for the 2% uprate relative to those used in the current analysis of record.

The use of the lower temperatures maximizes the critical mass flux in the Zaloudek correlation. The analysis uses the minimum composite RCS T_{HOT} , T_{COLD} and steam generator outlet temperatures that are calculated for the MUR conditions. The critical flow correlation used in the mass and energy releases for this analysis will provide an increase in the mass and energy release for a slightly lower fluid temperature.

Unit 1 and Unit 2 have been approved for leak before-break (LBB) methods (reference 14).^a This exemption eliminates the need to consider the dynamic effects associated with postulated RCS pipe breaks in the primary loop, eliminating the postulated primary system large pipe break from the subcompartment design basis. In addition, references 16 and 17 provide further justification for LBB application to eliminate surge line from the subcompartment design basis. Note, for Unit 2 only, reference 16 also approved LBB methods to eliminate, as a design basis, the accumulator and RHR piping. Therefore, the only break locations that need to be considered are the pressurizer spray line from the cold leg, the RHR line from the hot leg, and the smaller nozzles attached to the RCS.

Steam Generator Compartment

The licensing basis releases currently discussed in paragraph 6.2.1.2.3.2 for the steam generator compartment have been evaluated for the MUR uprate conditions. Based upon a comparison of the minimum hot full power RCS cold leg (T_{COLD}) temperature and the steam generator outlet temperature at the MUR uprate conditions, the temperature may be 0.7°F lower (537.6°F versus 538.3°F and 537.3°F versus 538.0°F) respectively, than that for the current licensing basis analysis. The minimum vessel outlet (T_{HOT}) temperature at the MUR uprate conditions is 603.8°F, which is 0.6°F higher (603.8°F versus 603.2°F). The differences in the temperatures presented are negligible and will have little effect on the mass and energy releases. In addition, based on the LBB exemption (reference 14), this exemption eliminates the need to consider the dynamic effects associated with postulated pipe break in the primary loop. In summary, the benefits of the decrease in mass and energy releases associated with the smaller primary system nozzle breaks, as compared to the larger RCS and larger RCS nozzle pipe breaks, more than offsets any penalty associated with possible increased releases which will result from decreased RCS coolant temperature. Therefore, the current licensing

^a The leak-before-break analyses have been evaluated as time-limited aging analyses (TLAA) for license renewal in accordance with 10 CFR Part 54. The results of this evaluation are provided in paragraph 19.4.6.1.

basis (mass and energy releases in table 6.2.1-27) remains bounding for the smaller nozzles attached to the RCS whenever the MUR uprate effects are considered.

Reactor Vessel Cavity

For the reactor vessel cavity, a RCS temperature decrease of 0.7°F for the vessel inlet will be seen with the MUR uprate conditions. This is the maximum difference between the basis for the current licensing basis and the MUR uprate data. This decrease in temperature is considered negligible. However, the current analysis includes 10% margin to account for residual uncertainties in the analytical model, noding effects, and initial reactor coolant conditions. It has been determined that the impact of the MUR is more than offset by the available 10% margin included in the current releases to account for residual uncertainties. Additionally, per reference 15, the average break opening area for the inlet break is 58 in.². The current releases for the reactor cavity break, which are based upon a 144 in.² break size, remain bounding for the subcompartment pressurization analysis considering the MUR uprate effects whenever the actual break size based upon the structural evaluation of the reactor pressure vessel is considered. Furthermore, per reference 14, Units 1 and 2 have been granted an exemption from a portion of the requirements of GDC 4 relative to LBB. This exemption eliminates the need to consider the dynamic effects associated with postulated pipe breaks in the primary loop. Therefore, due to the conservatisms in the calculations for table 6.2.1-26 releases, it can be concluded that the releases presented in table 6.2.1-26 remain bounding when the MUR conditions are considered.

Pressurizer Compartment

For the pressurizer compartment, there are two break locations of importance: the surge line and the spray line. For the surge line, the RCS temperature will increase by 0.6°F as a result of the MUR, which results in a decrease in releases. In addition, per references 16 and 17, Units 1 and 2 have been granted an exemption from a portion of the requirements of GDC 4. Based on the LBB exemption, this exemption eliminates the need to consider the dynamic effects associated with postulated rupture in the pressurizer surge line. For the spray line, the RCS temperature reduction was determined to be 0.7°F, which is also considered negligible. The current analysis includes 10% margin to account for uncertainties in the analytical model, noding effects, and initial reactor coolant conditions. It has been determined that the impact of the MUR is more than offset by the available 10% margin included in the current releases to account for residual uncertainties. In addition, comparing the pipe size assumed in the current analysis of record versus the as-built piping; therefore, it has been determined that the impact of the reduction in temperature is more than offset by the available margin in the break size assumed in the analysis. In summary, the releases in table 6.2.1-28A remain bounding for the spray line break for application to the subcompartment pressurization analysis.

- 6.2.1.2.3.3 <u>Initial Conditions for Subcompartment Pressure Analyses</u>. The conditions tabulated in tables 6.2.1-9, 6.2.1-11 through 6.2.1-16, and 6.2.1-25 and the assumptions in paragraph 6.2.1.2.3.2 are used as the initial conditions for the subcompartment pressure analyses.
- 6.2.1.2.3.4 <u>Flow Equation</u>. The flow equations used for calculating the sonic and subsonic flow between nodes are fully described in BN-TOP-4, Rev 1. These flow equations, as

applied in the program, are based on the homogeneous equilibrium model. In addition, the application of the equations to the Nuclear Regulatory Commission (NRC) benchmark problems and to the evaluation of the Batelle Frankfurt experiments has developed conservative differential pressure between nodes.

- 6.2.1.2.3.5 <u>Piping Systems</u>. RCS, main steam, and main feedwater piping were considered in the containment subcompartment analysis; however, the limiting transient for each subcompartment resulted from a break in the encompassed RCS piping except that the main feedwater line break is the limiting transient for steam generator subcompartments above el 220 ft. There are no flow restrictions in the RCS piping. The dimensions of the pipe are tabulated in table 5.4.3-1, and the break areas are tabulated in table 6.2.1-8. For the reactor cavity and pressurizer compartments, the breaks are at the pipe to vessel nozzle weld. The break location in the cold leg pipe does not impact the analysis of the steam generator compartment node pressures.
- 6.2.1.2.3.6 <u>Node Selection</u>. The nodalization for the reactor cavity, steam generator, and pressurizer compartments is performed so that nodal boundaries are at the location of flow obstructions or geometry changes within the compartment. These discontinuities create pressure differentials across nodal boundaries; i.e., between adjacent nodes. Within each node, there are no discontinuities and hence negligible pressure gradients.

The COPDA computer code, consistent with BN-TOP-4, Rev 1, assumes stagnation within each node at every calculational time step, and requires that nodal boundaries be taken at significant flow restrictions. This, as well as the other guidelines and recommendations of section 3.2 of NUREG-0609, is followed. In light of this, and the fact that COPDA is an NRC approved code, no sensitivity studies were performed. This is consistent with section 3.2.1 of NUREG-0609. Furthermore, sensitivity studies would require the addition of nodes that are not at discontinuities which violates the requirements of the COPDA code and would lead to meaningless results.

Assumptions on insulation behavior are given in paragraph 6.2.1.2.1.2 for the reactor cavity analysis. This is in accordance with section 3.2.2.3 of NUREG-0609. The steam generator and pressurizer compartment analyses include a reduction in flow area and subcompartment volume due to insulation around piping and vessels. Subcompartment nodalization is shown in figures 6.2.1-16, 6.2.1-17, and 6.2.1-24. The COPDA computer code input data is summarized in tables 6.2.1-9 through 6.2.1-28.

- 6.2.1.2.3.7 <u>Compartment Time Dependent Pressures</u>. The time dependent pressures for those nodes that determine the structural design of the compartment walls are shown in the graphs of figures 6.2.1-15, 6.2.1-18, 6.2.1-19, 6.2.1-20, 6.2.1-21, 6.2.1-22, 6.2.1-23, and 6.2.1-25.
- 6.2.1.2.3.8 <u>Vent Flowpath Flow Conditions</u>. The node and flow characteristics for each of the subcompartments are tabulated in tables 6.2.1-9 through 6.2.1-25. The time dependent mass and energy flow conditions are provided in tables 6.2.1-26, 6.2.1-27, and 6.2.1-28.

6.2.1.2.3.9 <u>Vent Flowpath Flow Coefficients</u>. There are two orifice coefficients, C_v and C_g : C_v is a viscous loss coefficient, and C_g is the ratio of <u>vena contract</u> area to aperture area. The quantity C discussed below is obtained by multiplying C_v and C_g . The orifice coefficient is a function of the Reynolds number; however, for sufficiently high Reynolds numbers (greater than 100,000), C becomes independent of the Reynolds number.

The definition of the Reynolds number is:

$$\frac{\text{Re}}{} = \frac{\text{SZ}}{\gamma}$$

where:

s = Velocity (ft/s).

Z = Characteristic length; i.e., diameter (ft).

 γ = Kinematic viscosity (ft²/s).

Kinematic viscosity (γ) can be expected to have values of around 10⁻⁴. Typical diameters of apertures are 10 to 30 ft. Velocities are 100 to 1000 ft/s. Therefore, <u>Re</u> will be very large and C will be independent of Re.

It is assumed throughout this section that flow coefficients obtained for single-phase flow are applicable for our two-phase situation and that the flow coefficients are constant with time.

The classification of openings and the development of head loss and flow coefficients are described in detail in BN-TOP-4, Rev 1.

6.2.1.3 <u>Mass and Energy Release Analysis for Postulated Loss-of-Coolant</u> Accidents

6.2.1.3.1 LOCA Mass and Energy Release Phases

The containment system receives mass and energy releases following a postulated rupture of the RCS. These releases continue through blowdown and post-blowdown.

The LOCA transient is typically divided into four phases:

- A. Blowdown which includes the period from accident initiation (when the reactor is at steady state operation) to the time that the RCS pressure reaches initial equilibrium with containment.
- B. Refill the period of time when the lower plenum is being filled by accumulator and safety injection water. At the end of blowdown, a large amount of water remains in the cold legs, downcomer, and lower plenum. To conservatively consider the refill period for the purpose of containment mass and energy releases, this water is instantaneously transferred to the lower plenum along with sufficient accumulator water to completely fill the lower plenum. This allows an uninterrupted release of mass and energy to containment. Thus, the refill period is conservatively neglected in the mass and energy release calculation.
- C. Reflood begins when the water from the lower plenum enters the core and ends when the core is completely quenched.

D. Post-Reflood (Froth) - describes the period following the reflood transient. For the pump suction break, a two-phase mixture exits the core, passes through the hot legs, and is superheated in the steam generators. After the broken loop steam generator cools, the break flow becomes two phase.

A discrepancy between volumetric heat capacities used in WCAP-10325-P-A (reference 10) and those documented in more recent ASME Code documents was identified. This condition was addressed in PWROG-17034-P-A (reference 18), where the NRC determined, with NSAL-06-6, NSAL-11-5, and NSAL-14-2 addressed, the continued use of WCAP-10325-P-A is acceptable for performing LOCA mass and energy release analysis for plants with large dry and sub-atmospheric containments.

6.2.1.3.2 Break Size and Location

Generic studies have been performed with respect to the effect on the LOCA mass and energy releases relative to postulated break size. The double-ended guillotine break has been found to be limiting due to larger mass flow rates during the blowdown phase of the transient. During the reflood and post-reflood phases, the break size has little effect on the releases.

Three distinct locations in the reactor coolant system loop can be postulated for pipe rupture:

- Hot leg (between vessel and steam generator).
- Cold leg (between pump and vessel).
- Pump suction (between steam generator and pump).

The break location analyzed and described herein is the double-ended pump suction guillotine break (10.48 ft²). Pump suction break mass and energy release have been calculated for the blowdown, reflood, and post-reflood phases of the LOCA.

The following information provides a discussion on each break location. The double ended hot leg guillotine has been shown in previous studies to result in the highest blowdown mass and energy release rates. Although the core flooding rate would be highest for this break location, the amount of energy released from the steam generator secondary side is minimal because the majority of the fluid which exits the core bypasses the steam generators in venting to containment. As a result, the reflood mass and energy releases are reduced significantly as compared to either the pump suction or cold leg break locations where the core exit mixture must pass through the steam generators before venting through the break.

For the hot leg break, there is no reflood peak as determined by generic studies (i.e., from the end of the blowdown period the releases would continually decrease). Therefore the reflood (and subsequent post-reflood) releases are not calculated for a hot leg break. The mass and energy releases for the hot leg break blowdown phase have been included in the scope of this containment integrity analysis.

The cold leg break location has also been found in previous studies to be much less limiting in terms of the overall containment peak pressure. The cold leg blowdown is faster than that of the pump suction break, and more mass is released into the containment. However, the core heat transfer is greatly reduced, and this results in a considerably lower energy release into containment. Studies have determined that the blowdown transient for the cold leg is, in general, less limiting than that for the pump suction break. During reflood, the flooding rate is greatly reduced and the energy release rate into the containment is reduced. Therefore, the cold leg break is not usually performed.

The pump suction break combines the effects of the relatively high core flooding rate, as in the hot leg break, and the addition of the stored energy in the steam generators. As a result, the pump suction break yields the highest energy flowrates during the post-blowdown period by including all of the available energy of the reactor coolant system in calculating the releases to containment. This break location has been determined to be the limiting break for typical dry containment plants. The choice of this break location for VEGP as the limiting break is consistent with other dry containment plants for the post blowdown phase of the event.

The analysis of the limiting break location for a dry containment has been performed. The double-ended pump suction guillotine break has historically been considered to be the limiting break location for the post blowdown phase of the event, by virtue of its consideration of all energy sources present in the RCS. The analyses support the conclusions of the double-ended pump suction (DEPS) as the limiting break case for the post-blowdown period, considering both the minimum and maximum safety injection cases. This break location provides a mechanism for the release of the available energy in the reactor coolant system, including both the broken and intact loop steam generators.

6.2.1.3.3 Application of Single Failure Criteria

An analysis of the effects of the single failure criteria has been performed on the mass and energy release rates for the DEPS break.

For the DEPS results an inherent assumption in the generation of the mass and energy release is that offsite power is lost. This results in the actuation of the emergency diesel generators required to power the safety injection system. This is not an issue for the blowdown period which is limited by the double-ended hot leg (DEHL) break.

Two cases have been analyzed for the effects of a single failure. The DEPS case with both minimum and maximum safety injection for the 3579 MWt rerated conditions was analyzed.

An evaluation has been performed to support the MUR power uprate.

The analysis of record presently assumes a NSSS power of 3650 MWt (102% of 3579 MWt), which includes a bounding allowance for the MUR power uprate in conjunction with a reduced calorimetric uncertainty.

The limiting case for the VEGP is the minimum safeguards case. This was determined by prior generic and specific VEGP analyses. In the case of minimum safeguards, the single failure postulated to occur is the loss of an emergency diesel generator. This results in the loss of one pumped safety injection train and the containment safeguards components on that diesel, thereby minimizing the safety injection flow. The analysis further considers the safety injection pump head curves to be degraded by 5%. (See Westinghouse Letter GP-15580 for discussion of flow margins.) This results in the greatest reduction possible for the emergency core cooling system (ECCS) components. For the case analyzing maximum safety injection, a conservative assumption was made due to the availability of only the diesel train failure criteria. The maximum safety injection flows were modeled with the minimum containment safeguard components available. This applies to heat exchangers, fan coolers, and the containment spray system. This assumption would provide a bounding assessment in maximizing the mass release but minimizing the heat removal capability.

6.2.1.3.4 Mass and Energy Release Data

- 6.2.1.3.4.1 <u>Significant Modeling Assumptions</u>. The following items ensure that the mass and energy releases are conservatively calculated for maximum containment pressure:
 - A. Maximum expected operating temperature of the reactor coolant system.
 - B. Allowance in temperature for instrument error and dead band (+6.0°F).
 - C. Margin in volume of 3% (which is composed of 1.6% allowance for thermal expansion and 1.4% for uncertainty).
 - D. NSSS power of 3650 MWt (102% of 3579 MWt), which includes a bounding allowance for the MUR power uprate in conjunction with a reduced calorimetric uncertainty.
 - E. Allowance for calorimetric error is included in item D.
 - F. Conservative coefficients of heat transfer (i.e., steam generator primary/secondary heat transfer and reactor coolant system metal heat transfer).
 - G. Allowance in core stored energy effect of fuel densification.
 - H. Margin in core stored energy (+15%).
 - I. Allowance for RCS pressure uncertainty (+50 psi).
- 6.2.1.3.4.2 <u>Blowdown Mass and Energy Release Data</u>. The SATAN-VI code is used for computing the blowdown transient and is the same as that used for the ECCS calculation in reference 11. The methodology for the use of this model is described in reference 10.

Tables 6.2.1-29 and 6.2.1-30 present the calculated mass and energy releases for the blowdown phase of the break analyzed for the DEPS and DEHL breaks, respectively. The mass and energy release for the DEPS break and the DEHL break, given in tables 6.2.1-29 and 6.2.1-30, terminate 22.0 and 25.0 seconds, respectively, after the initiation of the postulated accident.

6.2.1.3.4.3 <u>Reflood Mass and Energy Release Data</u>. The WREFLOOD code is used for computing the reflood transient and is a modified version of that used in the ECCS calculation in reference 11. The methodology for the use of this model is described in reference 10.

An exception to the mass and energy evaluation model described in reference 10 is taken, in that steam/water mixing in the broken loop has been included in this analysis. This assumption is justified, supported by test data, and summarized as follows:

The model assumes a complete mixing condition (i.e., thermal equilibrium) for the steam/water interaction. The complete mixing process, however, is made up of two distinct physical processes. The first is a two phase interaction with condensation of steam by cold injection water. The second is a single phase mixing of condensate and injection water. Since the mass and energy of the steam released is the most important influence to the containment pressure transient, the steam condensation part of the mixing process is the only part that need be considered. (Any spillage directly heats only the sump).

The most applicable steam/water mixing test data has been reviewed for validation of the containment integrity reflood steam/water mixing model. This data is that generated in 1/3 scale tests (reference 12), which are the largest scale data available and thus most closely simulate the flow regimes and gravitational effects that would occur in a PWR. These tests were designed specifically to study the steam/water interaction for PWR reflood conditions.

From the entire series of 1/3 scale tests, a group corresponds almost directly to containment integrity reflood conditions. The injection flow rates for this group cover all phases and mixing conditions calculated during the reflood transient. The data from these tests were reviewed and discussed in detail in reference 10. For all of these tests, the data clearly indicate the occurrence of very effective mixing with rapid steam condensation. The mixing model used in the containment integrity reflood calculation is therefore wholly supported by the 1/3 scale steam/water mixing data.

Additionally, the following justification is also noted. The limiting break for the containment integrity peak pressure analysis during the post-blowdown phase is the DEPS break. For this break, there are two flow paths available in the RCS by which mass and energy may be released to containment. One is through the outlet of the steam generator, the other via reverse flow through the reactor coolant pump. Steam, which is not condensed by ECCS injection in the intact RCS loops, passes around the downcomer and through the broken loop cold leg and pump in venting to containment. This steam also encounters ECCS injection water as it passes through the broken loop cold leg, complete mixing occurs, and a portion of it is condensed. It is this portion of steam which is condensed that is taken credit for in this analysis. This assumption is justified based upon the postulated break location and the actual physical presence of the ECCS injection nozzle. A description of the test and test results is contained in references 10 and 12.

The methodology previously discussed and described in reference 10 has been utilized and approved on the dockets for Catawba Units 1 and 2, McGuire Units 1 and 2, Sequoyah Units 1 and 2, Watts Bar Units 1 and 2, Millstone Unit 3, Beaver Valley Unit 2, and Surry Units 1 and 2.

Tables 6.2.1-31 and 6.2.1-32 present the calculated mass and energy release for the reflood phase of the DEPS break with minimum and maximum safety injection, respectively. A significantly higher discharge occurs during the period the accumulators are injecting (from 28.1 to 54.3 seconds for the minimum safety injection case and 28.1 to 54.1 seconds for maximum safety injection, as illustrated in tables 6.2.1-31 and 6.2.1-32).

The transient of the principal parameters during reflood is given in tables 6.2.1-33 and 6.2.1-34 for the minimum and maximum safety injection DEPS break cases.

6.2.1.3.4.4 <u>Post-Reflood Mass and Energy Release Data</u>. The FROTH code (reference 9) is used for computing the post-reflood transient. The methodology for the use of this model is described in reference 10. The mass and energy release rates calculated by the FROTH code are used in the containment analysis until the time of containment depressurization.

After depressurization, the mass and energy release from decay heat is based on the 1979 ANSI/ANS Standard, shown in reference 13, and the following input:

- A. Decay heat sources considered are fission product decay and heavy element decay of U-239 and Np-239.
- B. Decay heat power from fissioning isotopes other than U-235 is assumed to be identical to that of U-235.
- C. Fission rate is constant over the operating history of maximum power level.

- D. The factor accounting for neutron capture in fission products has been taken from Table 10 of ANSI/ANS 5.1-1979.
- E. Operation time before shutdown is 3 years.
- F. The total recoverable energy associated with one fission has been assumed to be 200 MeV/fission.
- G. Two sigma uncertainty (2 times the standard deviation) has been applied to the fission product decay.

Tables 6.2.1-35 and 6.2.1-36 present the two phase post-reflood mass and energy release data for the double-ended pump suction break minimum and maximum safety injection cases.

6.2.1.3.5 Sources of Mass and Energy

The sources of mass considered in the LOCA mass and energy release analysis are given in tables 6.2.1-37, 6.2.1-38, and 6.2.1-39. These sources are the reactor coolant system, accumulators, and pumped safety injection.

The energy inventories considered in the LOCA mass and energy release analysis are given in tables 6.2.1-40, 6.2.1-41, and 6.2.1-42. The energy sources include:

- Reactor coolant system water.
- Accumulator water.
- Pumped injection water.
- Decay heat.
- Core stored energy.
- Reactor coolant system metal.
- Steam generator metal.
- Steam generator secondary energy.
- Secondary transfer of energy (feedwater into and steam out of the steam generator secondary).

In the mass and energy release data presented, no Zirc-water reaction heat was considered because the clad temperature did not rise high enough for the rate of the Zirc-water reaction heat to be of any significance.

System parameters needed to perform confirmatory analyses are provided in table 6.2.1-43.

The consideration of the various energy sources in the mass and energy release analysis provides assurance that all available sources of energy have been included in this analysis. Thus, the review guidelines presented in Standard Review Plan Section 6.2.1.3 have been satisfied.

The mass and energy inventories are presented at the following times, as appropriate:

- Time zero (initial conditions).
- End of blowdown time.
- End of refill time.
- End of reflood time.
- Time of full depressurizations.
- End of analysis.

The methods and assumptions used to release the various energy sources are given in reference 10, except as noted in paragraph 6.2.1.3.4.3, which has been approved as a valid evaluation model by the Nuclear Regulatory Commission.

6.2.1.4 <u>Mass and Energy Release Analysis for Postulated Secondary System</u> Pipe Ruptures Inside Containment

Steam line ruptures occurring inside a reactor containment structure may result in significant releases of high-energy fluid to the containment environment, possibly resulting in high containment temperatures and pressures. The quantitative nature of the releases following a steam line rupture is dependent upon the many possible configurations of the plant steam system and containment designs as well as the plant operating conditions and the size of the rupture. These variations make a reasonable determination of the single absolute worst case for both containment pressure and temperature evaluations following a steam break difficult. This section describes the methods used in determining the containment responses to a variety of postulated pipe breaks encompassing wide variations in plant operation, safety system performance, and break size. The spectrum of breaks analyzed is listed in table 6.2.1-60.

6.2.1.4.1 Significant Parameters Affecting Steam Line Break Mass and Energy Releases

There are four major factors that influence the release of mass and energy following a steam line break: steam generator fluid inventory, primary-to-secondary heat transfer, protective system operation, and the state of the secondary fluid blowdown. The following is a list of those plant variables that determine the influence of each of these factors:

- A. Plant power level.
- B. Main feedwater system design.
- C. Auxiliary feedwater system design.
- D. Postulated break type, size, and location.
- E. Availability of offsite power.
- F. Safety system failures.
- G. Steam generator reverse heat transfer and RCS metal heat capacity.

The following is a discussion of each of these variables.

6.2.1.4.1.1 Plant Power Level. Steam line breaks can be postulated to occur with the plant in any operating condition ranging from hot shutdown to full power. Since steam generator mass decreases with increasing power level, breaks occurring at lower power generally result in a greater total mass release to the plant containment. However, because of increased energy storage in the primary plant, increased heat transfer in the steam generators, and the additional energy generation in the nuclear fuel, the energy release to the containment from breaks postulated to occur during power operation may be greater than for breaks occurring with the plant in a hot shutdown condition. Additionally, steam pressure and the dynamic conditions in the steam generators change with increasing power and have significant influence on both the rate of blowdown and the amount of moisture entrained in the fluid leaving the break following a steam break event.

Because of the opposing effects of changing power level on steam line break releases, no single power level can be identified as a worst case initial condition for a steam line break event. Therefore, several different power levels spanning the operating range as well as the hot shutdown condition have been analyzed.

6.2.1.4.1.2 <u>Main Feedwater System Design</u>. The rapid depressurization which occurs following a rupture may result in large amounts of water being added to the steam generators through the main feedwater system. Rapid closing isolation valves are provided in the main feedwater lines to limit this effect. Also, the piping layout downstream of the isolation valves affects the volume in the feedwater lines that cannot be isolated from the steam generators. As the steam generator pressure decreases, some of the fluid in this volume will flash into the steam generator, providing additional secondary fluid which may exit out the rupture.

The feedwater addition which occurs prior to closing of the feedwater line isolation valves influences the steam generator blowdown in several ways. First, the rapid addition increases the amount of entrained water in large-break cases by lowering the bulk quantity of the steam generator inventory. Secondly, because the water entering the steam generator is subcooled, it lowers the steam pressure, thereby reducing the flowrate out of the break. Finally, the increased flowrate causes an increase in the heat transfer rate from the primary-to-secondary system, resulting in greater energy being released out the break. Since these are competing effects on the total mass and energy release, no worst case feedwater transient can be defined for all plant conditions. In the results presented, the worst effects of each variable have been used. For example, moisture entrainment for each break is calculated assuming conservatively small feedwater additions so that the entrained water is minimized. Determination of total steam generator inventory, however, is based on conservatively large feedwater additions, as explained in paragraph 6.2.1.4.3.2.

The unisolated feedwater line volumes between the steam generator and the isolation valves serve as a source for additional high-energy fluid to be discharged through the pipe break. This volume is accounted for in the mass and energy release data presented in paragraph 6.2.1.4.3.2.

6.2.1.4.1.3 <u>Auxiliary Feedwater System Design</u>. Within the first minute following a steam line break, the auxiliary feed system is initiated on any one of several protection system signals. The addition of auxiliary feedwater to the steam generators increases the secondary mass available for release to the containment, as well as increases the heat transferred to the secondary fluid.

The effects of the steam generator mass are realistically and conservatively modelled in the calculation described in paragraph 6.2.1.4.3.2 by assuming full auxiliary feedwater flow to the faulted steam generator starting at time zero and continuing until manually stopped by the plant operator for the small break cases and zero power cases. The split break cases assume full auxiliary feedwater (AFW) flow to the faulted loop starting at the time of Hi-1 containment signal and the large double-ended rupture case assumes full AFW flow to the faulted steam generator starting at the time a safety injection or low steam generator mass reactor trip signal is received.

6.2.1.4.1.4 Postulated Break Type, Size, and Location.

A. Postulated Break Type

Two types of postulated pipe ruptures are considered in evaluating steam line breaks.

First is a split rupture in which a hole opens at some point on the side of the steam pipe or steam header but does not-result in a complete severance of the pipe. A single, distinct break area is fed uniformly by all steam generators until steam line isolation occurs. The blowdown flowrates from the individual steam generators are interdependent, since fluid coupling exists between all steam lines. Because flow limiting orifices are provided in each steam generator, the largest possible split rupture can have an effective area prior to isolation that is no greater than the throat area of the flow restrictor times the number of plant primary coolant loops. Following isolation, the effective break area for the steam generator with the broken line can be no greater than the flow restrictor throat area.

The second break type is the double-ended guillotine rupture in which the steam pipe is completely severed and the ends of the break displace from each other. Guillotine ruptures are characterized by two distinct break locations, each of equal area, but being fed by different steam generators. The largest possible guillotine rupture can have an effective area per steam generator no greater than the throat area of one steam line flow restrictor.

The type of break influences the mass and energy releases to containment by altering both the nature of the steam blowdown from the piping in the steam plant and the effective break area fed by each steam generator prior to steam line isolation. For example, a double-ended rupture in a pipe having a cross-sectional area of 2.4 ft² would appear as a 1.4-ft² rupture to a single steam generator feeding one end of the break but would appear as a 0.8-ft² rupture to each of steam generators feeding the other end of the break.

B. Postulated Size

Break area is also important when evaluating steam line breaks. It controls the rate of releases to the containment, as well as exerts significant influence on the steam pressure decay and the amount of entrained water in the blowdown flow. The data presented in this section include releases for four break areas at each of four initial power levels. Included are three double-ended and one split rupture, as follows.

1. A full double-ended pipe rupture downstream of the steam line flow restrictor. For this case, the actual break area equals the cross-sectional area of the steam line, but the blowdown from the steam generator with

the broken line is controlled by the flow restrictor throat area (1.4 ft²). The reverse flow from the intact steam generators is controlled by the smaller of the pipe cross section, the steam stop valve seat area, or the total flow restrictor throat area in the intact loops. The reverse flow has been conservatively assumed to be controlled by the flow restrictors in each of the intact loop steam generators. Actually, the combined flow from the three steam generators must pass through an 18-in. (1.42-ft²) line, which would greatly restrict the flow.

- 2. A small double-ended rupture at the steam generator nozzle having an area just larger than the area at which water entrainment occurs (i.e., with entrainment). Entrainment is assumed in the forward direction only. Dry steam blowdown is assumed to occur in the reverse direction.
- 3. A split break that represents the largest break which can neither generate a steam line isolation signal from the primary protection equipment nor result in moisture entrainment. Steam and feedwater line isolation signals are generated by high containment pressure signals for these cases. Being a split rupture, the effective area seen by the faulted steam generator increases by a factor of four, following steam line isolation. Conceivably, moisture entrainment could occur at that time. However, since steam line isolation for these breaks generally does not occur before 20 to 60 s, it is conservatively assumed that the pressure has decreased sufficiently in the affected steam generator to preclude any moisture carryover.
- 4. A small double-ended rupture at the steam generator nozzle having an area just smaller than that in which water entrainment occurs (i.e., without entrainment).

C. Postulated Break Location

Break location affects steam line blowdowns by virtue of the pressure losses which would occur in the length of piping between the steam generator and the break. The effect of the pressure loss is to reduce the effective break area- seen by the steam generator. Although this would reduce the rate of blowdown, it would not significantly change the total release of energy to the containment. Therefore, piping loss effects have been conservatively ignored in all blowdown results, except in the small double-ended ruptures in which moisture entrainment occurs. The effects of pipe friction are conservatively assumed to be sufficiently large in this case to prevent moisture entrainment in the reverse flow, thus minimizing water relief to the containment.

6.2.1.4.1.5 Availability of Offsite Power. The effects of the assumption of the availability of offsite power have been enveloped in the analysis. LOSP has been assumed where it delays the actuation of the containment heat removal systems (i.e., containment sprays and containment air coolers) due to the time required to start the emergency diesel generators. In these cases a 12-s diesel start time has been assumed. This includes the diesel initial sequencer loading step.

Offsite power has been assumed to be available where it maximizes the mass and energy released from the break due to:

- A. The continued operation of the reactor coolant pumps, which maximizes the energy transferred from the RCS to the steam generator.
- B. Continued operation of the feedwater pumps and actuation of the auxiliary feedwater system, which maximizes the steam generator inventories available for release.
- 6.2.1.4.1.6 <u>Safety System Failures</u>. In addition to assuming a LOSP, the following single active failures were considered.
 - Loss of one emergency diesel.
 - Failure of one main feedwater isolation valve.

The loss of one diesel results in the loss of one train of each of the containment heat removal systems. The analysis model conservatively accounted for the effects of the single failures by combining the failures into one bounding set of analyses. Two fast acting MSIVs are provided in each steam line. No more than one steam generator would experience an uncontrolled blowdown even if one of the MSIVs fails to close.

6.2.1.4.1.7 <u>Steam Generator Reverse Heat Transfer and Reactor Coolant System Metal Heat Capacity</u>. Once steam line isolation is complete, those steam generators in the intact steam loops become sources of energy that can be transferred to the steam generator with the broken line. This energy transfer occurs via the primary coolant. As the primary plant cools, the temperature of the coolant flowing in the steam generator tubes drops below the temperature of the secondary fluid in the intact units, resulting in energy being returned to the primary coolant. This energy is then available to be transferred to the steam generator with the broken steam line.

Similarly, the heat stored in the metal of the reactor coolant piping, the reactor vessel, and the reactor coolant pumps is transferred to the primary coolant as the plant cooldown progresses. This energy also is available to be transferred to the steam generator with the broken line.

The effects of both the RCS metal and the reverse steam generator heat transfer are included in the results presented in this document.

6.2.1.4.2 Description of Blowdown Model

A description of the blowdown model used is provided in reference 6. This reference is the basis for the data shown in tables 6.2.1-61 and 6.2.1-62.

6.2.1.4.3 Containment Response Analysis

The COCO computer code (reference 8), which is discussed in paragraph 6.2.1.1.3, was used to determine the containment responses following the postulated main steam line breaks (MSLB). The following assumptions were made to obtain these responses.

- 6.2.1.4.3.1 <u>Initial Conditions</u>. The initial containment conditions are the same as those used in the containment response analysis for the postulated RCS pipe ruptures. (See table 6.2.1-2.)
- 6.2.1.4.3.2 <u>Mass and Energy Release Data</u>. Using references 6 and 7 as a basis, mass and energy release data were developed to determine the containment pressure-temperature response for the spectrum of breaks analyzed. Tables 6.2.1-61 and 6.2.1-62 provide the mass and energy release data for the cases which result in the highest temperature and pressure. Table 6.2.1-64 provides specific plant data used for each case.

The rate of auxiliary feedwater addition represents the maximum runout flowrate to a fully depressurized steam generator. The value given for mass added by feedwater pumping assumes that no reduction in feedwater pump turbine speed occurs following a MSLB and prior to main feedwater isolation. Determination of feedwater flowrates prior to isolation assumed that the feedwater regulating valve in the broken loop goes wide open while those in the intact loops remain in their prebreak positions. Actual isolation is dependent on signals generated by the primary protection system. Feedwater isolation for the split breaks was based on the time required to reach the containment pressure setpoint that generates the isolation signal. For the split breaks and small double-ended rupture, feedwater was conservatively assumed to match the steam flow until the time of feedwater isolation.

6.2.1.4.3.3 Containment Pressure-Temperature Results. Figures 6.2.1-26 through 6.2.1-29 provide curves of the resultant containment pressure-temperature analyses for the cases producing the highest peak containment pressure and temperature. Table 6.2.1-65 summarizes the results of all the cases analyzed and indicates the times at which dryout occurs and the various containment pressure setpoints are reached. The sequence of events following a postulated MSLB is listed in tables 6.2.1-66 and 6.2.1-67 for worst pressure and temperature cases.

As illustrated in figure 6.2.1-26, case 16 results in a peak pressure of 36.5 psig. This case represents the peak calculated containment pressure for the spectrum of breaks analyzed. The containment vapor temperature profile versus time for this case is provided in figure 6.2.1-27.

It is important to note that the peak calculated pressure is coincident with the completion of the blowdown of the contents of the affected steam generator. In all cases, the peak calculated containment pressure demonstrates considerable margin below the containment design pressure.

As illustrated in figure 6.2.1-29, case 13 results in a peak vapor temperature of 303.1°F. This case represents the peak calculated containment vapor temperature for the spectrum of breaks analyzed. The containment pressure profile versus time for this case is provided in figure 6.2.1-28.

6.2.1.5 <u>Minimum Containment Pressure Analysis for Performance Capability</u> Studies on Emergency Core Cooling System

The containment backpressure and temperature and the containment wall-condensing heat transfer coefficient, used for the limiting case CD = 0.6 double-ended cold leg guillotine break for the (ECCS) analysis found in subsection 15.6.5, are presented in figures 6.2.1-30, 31, and 32. The containment backpressure is calculated using the methods and assumptions described

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in Westinghouse-Emergency Core Cooling System Evaluation Model -Summary, WCAP-8339, Appendix A. Input parameters including the containment initial conditions, net free containment volume, passive heat sink materials, thicknesses, surface areas, starting time, and number of containment cooling systems used in the analysis are described below.

6.2.1.5.1 Mass and Energy Release Data

The break mass and energy releases to containment during the blowdown and reflood portions of the limiting break transient are presented in tables 6.2.1-68 and 6.2.1-69, respectively.

The mathematical models which calculate the mass and energy releases to the containment are described in subsection 15.6.5. Since the requirements of Appendix K of 10 CFR 50 are very specific in regard to the modeling of the RCS during blowdown and since the models used are in conformance with Appendix K, no alterations to those models have been made in regard to the mass and energy releases. A break spectrum analysis is performed. (See the double-ended cold leg guillotines which affect the mass and energy released to the containment.) This effect is considered for each case analyzed. During refill, the mass and energy released to the containment is assumed to be zero, which minimizes the containment pressure. During reflood, the effect of steam-water mixing between the safety injection water and the steam flowing through the RCS intact loops reduces the available energy released to the containment vapor space and, therefore, tends to minimize containment pressure.

6.2.1.5.2 Initial Containment Internal Conditions

The following initial values were used in the analysis:

Containment pressure (psia)	14.7
Containment temperature (°F)	90
Refueling water storage tank temperature (°F)	40
NSCW temperature (°F)	40
Outside temperature (°F)	17

The containment initial conditions of 90°F and 14.7 psia are representatively low values anticipated during normal full-power operation. The initial relative humidity was conservatively assumed to be 99%.

6.2.1.5.3 Containment Volume

The volume used in the analysis is $2.95 \times 10^6 \text{ ft}^3$ plus an additional amount to incorporate the effect of containment purge, resulting in a total containment volume of $3.20 \times 10^6 \text{ ft}^3$.

6.2.1.5.4 Active Heat Sinks

The containment spray system and the containment fan coolers operate to remove heat from the containment. Pertinent data for these systems which were used in the analysis are presented in table 6.2.1-71. The heat removal capability of each fan cooler is presented in table 6.2.2-2.

Because the fan coolers use nuclear service cooling water (NSCW), the lowest normal NSCW temperature (40°F) was used in the analysis.

The containment sump temperature was not used in the analysis because the maximum peak cladding temperature occurs prior to initiation of the recirculation mode for the containment spray system. In addition, heat transfer between the sump water and the containment vapor space was not considered in the analysis.

6.2.1.5.5 Steam-Water Mixing

Water spillage rates from the broken loop accumulator are presented in table 6.2.1-70.

6.2.1.5.6 Passive Heat Sinks

The passive heat sinks used in the analysis, with their thermophysical properties, are given in table 6.2.1-72. The passive heat sinks and thermophysical properties were divided in compliance with Branch Technical Position CSB6-1, Minimum Containment Pressure Model for Pressurized-Water Reactor (PWR) ECCS Performance Evaluation.

6.2.1.5.7 Heat Transfer to Passive Heat Sinks

The condensing heat transfer coefficients used for heat transfer to the steel containment structures are given in figure 6.2.1-32 for the limiting break. The containment temperature transient for the limiting break is shown in figure 6.2.1-31.

6.2.1.5.8 Other Parameters

Operating containment purge at the onset of the double-ended cold leg guillotine break was shown to cause a drop in containment pressure of less than 0.19 psi. An increase in the containment volume was incorporated into the analysis, as discussed in paragraph 6.2.1.5.3. No other parameters have a substantial effect on the minimum containment pressure analysis.

6.2.1.5.9 Standard Review Plan Evaluation

The VEGP does not employ the heat transfer coefficients supplied in the Standard Review Plan.

The heat transfer coefficients were calculated in conformance with WCAP-8339, Appendix A, which has received Nuclear Regulatory Commission approval. This reference is for the COCO code and predates CSB6-1. Results using the heat transfer coefficients have been found acceptable in other plant applications.

6.2.1.6 <u>Instrumentation Requirements</u>

Adequate instrumentation is provided to monitor the conditions inside the containment and actuate the appropriate engineered safety features, should those conditions exceed the predetermined levels. The instruments measure the containment pressure, containment

atmosphere radioactivity, purge exhaust effluent radioactivity, and containment hydrogen concentration.

The containment pressure is measured by four independent Q-class pressure transmitters and fed into the engineered safety features actuation system (ESFAS) as described in subsection 7.3.1. Upon detection of excessively high pressure inside the containment, the appropriate safety actuation signals are generated which automatically activate the necessary safety systems. These physically separated pressure transmitters are located outside the containment and connected to their sensors by filled and sealed hydraulic lines. Refer to section 7.3 for a detailed description.

The containment atmosphere radiation level is monitored by four independent Q-class area monitors located at the operating deck inside the containment building. The measurements of monitors RE-0002 and RE-0003 are continuously fed into the ESFAS logic and, in modes 1 through 4, cause the actuation of containment ventilation isolation (CVI) safety signals, should the measured radiation levels exceed their setpoints. In mode 6 during core alterations or movement of irradiated fuel assemblies in containment, two channels of radiation monitors are required operable to provide input to control room alarms to ensure prompt operator action to manually close the containment purge and exhaust valves. In addition, the containment purge exhaust air radiation is measured by a non-Q-class, three-channel airborne effluent monitor located outside the containment in the purge exhaust duct. Its isolated output is also fed into the ESFAS logic and, in modes 1 through 4, causes actuation of the CVI signal when the monitored radiation level exceeds the setpoint. In mode 6 during core alterations or movement of irradiated fuel assemblies, only operable radiation monitors are required to alert the operators of the need for containment ventilation isolation. Manual isolation using individual valve hand switches following a radiation alarm is the means for isolating containment in the event of a fuel handling accident during shutdown. For detailed information on the containment area radiation monitors, effluent monitor, and the ESFAS operation, refer to subsections 12.3.4 and 11.5.2 and section 7.3, respectively.

The containment hydrogen concentration is measured by two redundant non-Q-class hydrogen monitors as described in subsection 6.2.5. The readouts and alarms are provided in the control room to facilitate manual actuation of safety-related hydrogen control systems by the operator, should it become necessary.

6.2.1.7 References

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- 18. PWROG-17034-P-A, "Evaluation of the WCAP-10325-P-A Westinghouse LOCA Mass & Energy Release Methodology," Revision 0, March 2020.

6.2.2 CONTAINMENT HEAT REMOVAL SYSTEMS

The functional performance objective of the containment heat removal system as an engineered safety features (ESF) system is to reduce the containment temperature and pressure following a loss-of-coolant accident (LOCA) or main steam line break (MSLB) accident inside the containment by removing thermal energy from the containment atmosphere. These cooling systems also serve to limit offsite radiation levels by reducing the pressure differential between the containment atmosphere and the external environment, thereby diminishing the driving force for leakage of fission products from the containment to the atmosphere. The containment heat removal systems include the containment cooling system and the containment spray system. The containment cooling system described here also functions during normal operation to maintain a suitable atmosphere for the equipment located within the containment.

6.2.2.1 <u>Containment Cooling System</u>

6.2.2.1.1 Design Bases

6.2.2.1.1.1 <u>Safety Design Bases.</u>

- A. The containment cooling system is designed to withstand the effects of natural phenomena such as earthquakes, winds, tornadoes, or floods.
- B. The containment cooling system is automatically placed in operation on receipt of a safety injection (SI) signal following a LOCA or MSLB accident.
- C. The containment cooling system is designed so that a single failure of any active component, assuming loss of offsite power, cannot impair the capability of the system to perform its safety function.
 - The capability of isolating components, systems, or piping is provided, if required, so that the system's safety function will not be compromised.
- D. Active components of the containment cooling system are capable of being tested during plant operation. Provisions are made for inspection of major components at appropriate times specified in American Society of Mechanical Engineers (ASME) Boiler and Pressure Vessel Code Section XI.
- E. The containment cooling system components required to mitigate the consequences of an accident are designed to remain functional in the accident environment and to withstand the dynamic effect of the accident.
- F. The containment cooling system in conjunction with the containment spray system is capable of removing sufficient thermal energy and subsequent decay heat from the containment atmosphere following the postulated LOCA or MSLB accident to maintain the containment pressure below design values.
- G. The containment cooling system is designed and fabricated to codes consistent with Regulatory Guide 1.26 as described in table 3.2.2-1 and Seismic Category 1 in accordance with Regulatory Guide 1.29. The power supply and control functions are in accordance with Regulatory Guide 1.32.
- 6.2.2.1.1.2 <u>Power Generation Design Bases</u>. The containment cooling system is designed to limit the ambient containment air temperature during normal plant operation to 120°F with any four of the eight coolers operating. Normal operation and the power generation design bases are discussed in section 9.4.

6.2.2.1.2 System Design

6.2.2.1.2.1 <u>General Description</u>. The containment cooling units are designed to the codes and standards identified in table 3.2.2-1; flood design is discussed in section 3.4; missile protection is discussed in section 3.5. Protection against dynamic effects associated with the

postulated rupture of piping is discussed in section 3.6. Environmental design and equipment qualification is discussed in section 3.11. The actuation system is discussed in section 7.3.

- 6.2.2.1.2.2 System Description. The containment cooling system consists of eight separate 25% fan cooler units inside the containment. The system piping and instrumentation diagram is given in drawing 1X4DB212. The location and arrangement of the fan coolers are indicated in drawings 1X4DJ4103, 1X4DJ4113, 1X4DJ4123 and 1X4DJ4133. The system is designed to permit periodic testing and inspection as discussed in paragraph 6.2.2.1.4. Table 6.2.2-1 provides a tabulation of the design and performance data for each containment cooling unit. The containment fan coolers reject heat to the nuclear service cooling water (NSCW) system, which is described in subsection 9.2.1.
- 6.2.2.1.2.3 Component Description. Each 25%-capacity fan cooling unit consists of a vane axial fan, a fan motor, copper-nickel cooling coils with copper fins, a carbon steel housing, round metal ducting, and a concrete discharge duct. The 8 units are located at el 238 ft in the containment building and discharge to the bottom of the containment building through four concrete ducts. Each fan unit has two speeds of operation, high speed for normal operation and low speed for post-accident operation. For purposes of heat removal from the containment atmosphere, any combination of four fans in slow speed is sufficient.

The containment cooling system is separated into two trains consisting of four fan cooler units each. The two trains are supplied cooling water and electrical power from the corresponding train of the NSCW system and the Class 1E electrical power system.

Each concrete air supply duct contains three large outlets located above the maximum calculated containment flood elevation to supply cool air to the containment. The air cooling unit enclosures and concrete air supply duct remain intact following a design basis accident (DBA). There is a backdraft damper located at the inlet of each concrete air supply duct to prevent overpressurizing the ducting and coolers. The capability to remain intact is discussed in relation to the hydrogen mixing function of the containment cooling system in paragraph 6.2.5.2.1. Plan and elevation drawings of the containment showing the routing of airflow guidance ductwork are given in drawings 1X4DJ4103, 1X4DJ4113, 1X4DJ4123 and 1X4DJ4133. The fans and motors are designed to operate in the containment post-accident environment. The heat removal capability of the containment fan coolers versus containment temperature for the maximum NSCW inlet temperature is provided in table 6.2.2-2.

6.2.2.1.2.4 System Operation. The containment cooling system is an ESF system that is in use during normal plant operation with four fans operating at high speed. The containment isolation valves on the NSCW line to the containment air cooling units are normally open. System ESF operation is initiated automatically upon receipt of an SI signal. Upon receipt of an SI signal, all fans are restarted at low speed. The basis for the setpoint for the automatic initiation of the cooling system is the diesel start time (12 + 0.5 s) and the load sequencer for the fan coolers (30.5 s). The containment cooling system is fully operational within 40.5 s following receipt of an SI signal. The containment air cooling units can be stopped and started from the control room and from the shutdown panels.

6.2.2.1.3 Safety Evaluation

- A. The safety-related portions of the containment cooling system are located in the containment building. This building is designed to withstand the effects of natural phenomena such as earthquakes, winds, tornadoes, or floods.
- B. Operation of the containment cooling system is initiated automatically following the receipt of an SI signal. Use of this signal provides a reliable indication that a LOCA or a MSLB accident has occurred inside containment. Operation of the containment air coolers may also be manually initiated from the control room and from the shutdown panels. A detailed description of the actuation system is contained in section 7.3.
- C. Two trains, each containing four air cooling units, are supplied from redundant emergency power sources and redundant NSCW trains. Failure of any component in one train will not affect the operability of the other train. The containment analyses of LOCA and MSLB accidents were performed in subsection 6.2.1, assuming the availability of one of two containment air cooling trains. A failure modes and effects analysis of the containment cooling system is presented in table 6.2.2-3.
- D. Capability is provided to periodically test the entire startup sequence of the containment air cooling system. Active components can be tested periodically during plant operation to verify operability. The entire system can be inspected during unit shutdown. Additional information is contained in section 3.1, paragraph 6.2.2.2.4, and the Technical Specifications.
- E. The containment air cooling units are tested and demonstrated to perform in a simulated MSLB and LOCA environment. The units are located in a manner to minimize the effects of jet impingement and pipe whip in case of a high-energy line break.
- F. The analyses show that the containment cooling system in conjunction with the containment spray system is capable of removing sufficient heat energy and subsequent decay heat from the containment atmosphere to ensure the accident peak pressure is below the containment design pressure. Accident analyses assume the occurrence of a single failure that results in the loss of one air cooling train and one containment spray train. Containment cooler heat removal capacity is provided in table 6.2.2-2.

Containment accident analysis assumes a constant NSCW temperature equal to the highest anticipated system temperature (95°F) to maximize the calculated containment peak pressure. The assumptions used in calculating this temperature are discussed in subsections 9.2.1 and 9.2.5.

Curves showing heat removal rates of the containment air cooling system, and containment total pressure, and temperatures as a function of time for minimum ESF performance are given in the figures of subsection 6.2.1.

Location of the containment air cooling units in different quadrants of the containment, the difference in elevation between suction and discharge points, and the significant flowrate developed (table 6.2.2-1) ensure adequate circulation in the containment following a LOCA which prevents the formation of localized high temperature air pockets or areas of high combustible gas concentration. The mixing capability of this system is supplemented by the containment spray

- system and natural convection. Additional information is contained in subsection 6.2.5 and section 9.4.
- G. The containment cooling system is designed to Seismic Category 1 requirements as specified in section 3.2.

6.2.2.1.4 Testing and Inspection

Fans are tested and rated by the manufacturer in accordance with the standards of the Air Moving and Conditioning Association.

The containment air cooling units are tested and/or analyzed by the manufacturer to ensure operation in the post-accident environmental conditions indicated in section 3.11 and following a safe shutdown earthquake (SSE).

The preoperational testing of the containment air cooling system, as well as its components, demonstrates the initial capability of the equipment. Written test procedures establish minimum acceptance values for all tests. See section 14.2 for details of the preoperational test program.

Following completion of the preoperational integrated leakage rate tests, fans are operated at reduced speed, and the motor currents are monitored with temporarily attached ammeters to verify satisfactory operation in denser than normal atmospheres.

The fans are run and monitored during plant shutdown in accordance with the planned maintenance program.

6.2.2.1.5 Instrumentation Requirements

Containment cooling system controls and instrumentation are discussed in sections 7.3 and 7.5, respectively. The system is designed to function automatically following receipt of an SI signal. Fans and cooling water supply can also be controlled remotely from the control room and from the shutdown panels.

The status of the containment cooling system is displayed in the control room. The NSCW and emergency power supplies are discussed in subsection 9.2.1 and section 8.3, respectively.

6.2.2.2 Containment Spray System

6.2.2.2.1 Design Bases

6.2.2.2.1.1 Safety Design Bases.

- A. The containment spray system is designed to withstand the effects of natural phenomena such as earthquakes.
- B. The containment spray system is automatically placed in operation on receipt of two out of four containment pressure (high-3) signals.

- C. The containment spray system is designed so that a single failure of any active component, assuming loss of offsite power, cannot impair the capability of the system to perform its safety function during the injection phase.
 - A single active or passive failure cannot impair the capability of the system to perform its safety function during the recirculation phase.
- D. Active components of the containment spray system are capable of being tested during plant operation. Provisions are made for inspection of major components at appropriate times specified in ASME Boiler and Pressure Vessel Code, Section XI.
- E. The containment spray system components are designed to remain functional during the accident environment and to withstand the dynamic effect of the accident.
- F. The containment spray system in conjunction with the containment cooling system is capable of removing sufficient thermal energy and subsequent decay heat from the containment atmosphere following the postulated LOCA or MSLB accident to maintain the containment pressure below design values.
- G. The containment spray system is designed and fabricated to codes consistent with Regulatory Guide 1.26 as described in table 3.2.2-1 and Seismic Category 1 in accordance with Regulatory Guide 1.29. The power supply and control functions are in accordance with Regulatory Guide 1.32.
- H. The design basis for the CSS with regard to the effects of debris on the emergency sump strainers is a risk-informed analysis, which shows the risk associated with the effects of debris is very small as defined by Regulatory Guide 1.174. The conclusion is based on plant-specific testing and analyses using inputs and assumptions that provide safety margin and defense-in-depth.

Details of the design basis for the effects of debris on the function of the emergency sump strainers are provided in FSAR Appendix 6A.

6.2.2.2.1.2 <u>Power Generation Design Bases</u>. The containment spray system has no power generation design bases.

6.2.2.2.2 System Design

- 6.2.2.2.2.1 <u>General Description</u>. The containment spray system is designed to the codes and standards identified in table 3.2.2-1; flood design is discussed in section 3.4; missile protection is discussed in section 3.5. Protection against dynamic effects associated with the postulated rupture of piping is discussed in section 3.6. Environmental design and equipment qualification is discussed in section 3.11. The actuation system is discussed in section 7.3.
- 6.2.2.2.2.2 <u>System Description</u>. The containment spray system, shown schematically in drawing 1X4DB131, consists of two pumps, spray ring headers and spray nozzles, valves, and connecting piping. Initially, water from the refueling water storage tank (RWST) is used for the

containment spray followed by water recirculated from the containment emergency sump. The recirculated spray is mixed with trisodium phosphate in the containment sump region.

At the RWST empty level alarm the operator should initiate manual switchover of the containment spray pumps to the recirculation mode of operation. See table 6.3.2-7 for a summary of the necessary manual actions for switchover. Adequate transfer allowance is provided to allow the operator to perform the switchover sequence without securing the containment spray pumps. The total amount of borated refueling water injected into the containment by the charging, safety injection, residual heat removal, and containment spray pumps will provide a sump pH of 7.0 or above when mixed with the contents of the trisodium phosphate baskets.

A single failure can occur which may prevent the switchover of one of the two trains. However, a single failure cannot prevent the switchover of both trains simultaneously. The containment pressure transient analysis shows that only one of the two redundant spray trains is necessary to prevent containment pressure from reaching the containment design point. Thus, even if one train is not available following the switchover, the remaining operating train is sufficient to control containment pressure, assuming that four of the eight containment fan coolers are also in operation.

A gas accumulation monitoring and trending process for the Vogtle Unit 1 and 2 ECCS and containment spray systems has been established to meet the requirements of NRC Generic Letter 2008-01.

6.2.2.2.3 <u>Component Description</u>. The mechanical components of the containment spray system are described in this section. Component design parameters are given in table 6.2.2-4. Parts of the system in contact with borated water are stainless steel or an equivalent corrosion-resistant material.

Corrosion tests have been performed on the materials that the spray would come in contact with, e.g., the paint on the inside of the containment structure. (Tests are detailed in WCAP-7825 and NUREG CR-3803.) These tests have shown that no significant amount of corrosion products is produced. Those corrosion products or any chemical precipitation of appreciable size that does occur is trapped by the sump filter screen. The screen size is smaller than the line piping, residual heat removal heat exchanger tubes, and the spray nozzles, so that particles which could potentially block the system will be filtered out. The spray nozzle material (stainless steel, SA351) was chosen for its resistance to corrosion. Tests have been performed on this material in the same type of environment that the nozzle would see during spray actuation. (Corrosion tests of austenitic stainless steel are detailed in WCAP-7803 and WCAP-11611.) The resulting corrosion levels were very low.

Stress corrosion does not present a problem since it only becomes a factor under a combination of conditions of high stress levels at high temperatures for extended periods of time. The stress levels in the pumps during operation are relatively low; the working temperature of the pumps is less than 212°F.

The pumps are located outside the containment. The external surfaces of the pumps, as well as the pump motors, are not subjected to the corrosive atmosphere of the spray solution. Only the internal stainless steel surfaces of the pumps are exposed to a corrosive atmosphere.

6.2.2.2.3.1 Refueling Water Storage Tank. This tank serves as a source of emergency borated cooling water for injection and containment spray. It is normally used to fill the refueling canal for refueling operations. During all other plant

operating periods, it is aligned to the suction of the emergency core cooling pumps and the containment spray pumps. The tank is a concrete tank lined with type 304 stainless steel plates. The nominal tank volume is 715,000 gal. The contents of this tank are protected from freezing by a sludge mixing system which includes an electric circulation heater. Lines and appurtenances to the RWST serving a safety-related function are heat traced as necessary to prevent freezing.

6.2.2.2.3.2 Containment Spray Pumps. The containment spray pumps are of the horizontal centrifugal type, driven by electric motors powered from the emergency buses.

The design head of the pumps is sufficient to continue at rated capacity with a minimum level in the RWST against a head equivalent to the sum of the design pressure of the containment, the head to the uppermost nozzles, line losses, and nozzle pressure losses. The containment spray system is designed so that adequate net positive suction head (NPSH) is provided to the containment spray pumps, in accordance with Regulatory Guide 1.1.

The recirculation mode of operation at -0.3 psig containment pressure and 211°F sump temperature results in the limiting NPSH.

The adequacy of the available NPSH was evaluated as described in Appendix 6A.

Design parameters for these pumps are presented in table 6.2.2-4.

6.2.2.2.3.3 Spray Nozzles. The hollow cone spray nozzles are not subject to clogging by particles less than 1/4 in. in size and produce a drop size spectrum with a mean diameter of less than 700 mm at 40 psi differential pressure. During spray recirculation operation, the water is screened through perforated plates with 3/32-in. diameter holes (a small percentage – 124 holes – are larger than 3/32-in. diameter but < 1/4-in. diameter – see section 6.1.2.) before leaving the containment emergency sump. With the spray pump operating at design conditions and the containment at design pressure, the pressure drop provided across the nozzles exceeds 40 psi. The spray nozzles used in the construction of the containment spray system are designed to withstand differential pressures in excess of those expected to occur as a result of an accident.

The spray nozzles are stainless steel and have a 3/8-in.-diameter orifice.

- 6.2.2.2.3.4 Spray Additive Tank. The spray additive tank has been abandoned in place.
- 6.2.2.2.3.5 Spray Additive Eductors. The spray additive eductors serve only to maintain the spray system pressure boundary integrity.
- 6.2.2.2.3.6 Piping, Valves, and Containment Sumps. The piping used in the construction of the containment spray system is designed to withstand differential pressures in excess of those expected to occur as a result of an accident. The containment sump recirculation lines that run from the containment sump to the containment spray pumps are enclosed within guard pipes from the containment emergency sump floor to the first valve outside the containment. The pipe and guard pipe are capable of withstanding containment pressure and temperature. The guard pipe prevents leakage from the containment if the recirculation line ruptures.

The containment sumps are designed in accordance with the requirements of Regulatory Guide 1.82, as described in Appendix 6A. The risk-informed methodology, applied to evaluate the risk associated with effects of accident-generated debris, shows that the increase in risk associated with debris that would exceed the design limits of the sump strainers is very small, in accordance with the acceptance criteria of Regulatory Guide 1.174. These sumps are located in a manner that protects them from the effects of high-energy line breaks, and they are separated from each other. The elevation of the containment emergency sumps are selected to allow optimum use of the available coolant. The sump intakes are protected by vertically stacked disk screens. The size of the opening in the screens is based on the minimum flow area through the components that receive coolant from the emergency sumps.

Analyses were conducted to ensure that effects such as reduction of NPSH and screen blockage will not result in degraded pump or system performance. The screens are installed in a manner to facilitate inspection of the structures and pump suction intakes.

There are two Containment Spray (CS) sumps. A screen is installed on each sump. The CS screens are composed of four stacks of 14 disks that are 30-in. long by 30-in. wide by ~40-in. high, four of which provide 590 ft² of perforated plate area and 133 ft² of circumscribed surface area per sump. Each typical screen disk is a welded assembly of two perforated plates and their structural support components. The plate-hole (perforation) diameter of the screen is 3/32 in. (a small percentage - 124 - holes are larger than 3/32-in.diameter but none are larger than ½-in. diameter – see section 6.1.2.) The screen is mounted over the containment sump. Any particles that penetrate the screen will pass through the piping pumps and valves as well as the 3/8-in. diameter containment spray nozzle openings without difficulty. The screens bolt to the floor and may be removed by unbolting individual screen sections for inspection during shutdown periods, or the dedicated inspection port may be used.

Water sprayed in the refueling canal from the containment sprays may escape back to the elevation of the emergency sump through two 12-in. drain pipes located at the lowest point of the refueling canal. The water passes from the canal to a passageway on the containment floor. Spray water from 33 out of 171 nozzles during 1-train operation or a maximum of 66 out of a total of 342 nozzles from 2 containment spray trains in operation falls into the refueling canal for a maximum canal fill rate of approximately 500 gal/min and 1000 gal/min, respectively. This represents less than 20% of the total spray rate in either case.

The drain layout is such that each 12-in. drain line is capable of passing approximately 2000 gal/min; therefore, there is no danger of starving the sump via the refueling canal. The drain piping is isolated during refueling and left open during normal reactor operation. Plant refueling procedures ensure that these drain pipes are opened after refueling prior to plant startup.

One containment spray pump is provided for each train. A single failure therefore leaves one of the two trains in service. The containment spray pumps are located within compartments sealed by watertight doors; a postulated rupture in one train cannot flood the other.

The first of the two motor-operated valves in series on the recirculation lines are totally enclosed in protective chambers, ensuring that all liquid escaping from a damaged or leaky valve does not escape to the outer environment. All materials that can come in contact with recirculation fluid are austenitic stainless steel. The spray headers are located in the proximity of the containment liner of the dome. The spray headers are anchored to the concrete through the liner plate. Each system is designed for SSE and appropriate thermal and dead weight loading conditions. The spray system is designed for maximum coverage in the containment, with the nozzles located and oriented so that the spray will not be blocked by structures or equipment.

- 6.2.2.2.3.7 Motors for Pumps and Valves. The motors for the containment spray system components will be designed in accordance with specifications discussed for the motors in the SI system. (See section 6.3.)
- 6.2.2.2.3.8 Electrical Supply. Details of the emergency bus power sources are discussed in chapter 8.
- 6.2.2.2.4 <u>System Operation</u>. The spray system is actuated by a signal initiated manually from the control room or automatically on coincidence of two of four containment pressure (high-3) signals. These signals start the containment spray pumps and open the discharge valves to the spray headers.

During all modes of operation except refueling, the suction of the pumps is normally aligned to the RWST. The spray pumps continue to draw a suction on the RWST until the later stages of the injection phase. When the RWST level reaches empty and the ECCS is realigned from the injection to recirculation, the spray pump suction is remote-manually shifted to the containment emergency sumps. This must be complete by the time the RWST level reaches 2.515 ft.

6.2.2.2.5 <u>pH Control</u>. Containment spray could be operated for a short period of time and recirculation will never occur. This is during the secondary line break or inadvertent containment spray actuation events. Since the containment spray can be operated only in the injection phase without any recirculation, the spray will be acidic (4.5 pH). If containment spray is actuated and terminated prior to recirculation, a controlled cleanup and inspection of equipment in containment should begin within 5 days of the event. This is necessary to assess and recover from the effects of the acidic containment sprays on equipment in containment prior to reaching unacceptable conditions.

The initial containment spray pH will be approximately 4.5 during the injection phase which can last up to approximately 4.5 hours. After switchover to the recirculation phase, the spray pH will be between 7.0 and 10.5. If the containment spray system is actuated during a LOCA, its operation should continue in the injection phase and through switchover and into the recirculation phase. Operation of the containment spray system in the recirculation mode is necessary to ensure that the containment is resprayed with the solution (7.0 to 10.5 pH) from the emergency sumps. Once this respray occurs, then the containment sprays can be terminated, provided that containment pressures have already decreased to acceptable levels.

The pH of the sump solution will be adjusted to greater than 7.0 within 8 hours of the accident (LOCA) initiation to counteract the buildup of chloride concentrations to critical levels. This is considered a conservatively short period in which to make the pH adjustment, even with the potential for rapidly increasing sump chloride concentrations. The materials in containment are qualified for long term exposure to a high pH solution and will not be adversely affected by short term exposure to a low pH solution.

A summary of procedural requirements for containment spray operation follows:

- A. If containment spray is actuated and terminated prior to recirculation, a controlled cleanup and inspection of equipment in containment should begin within 5 days of the event.
- B. If containment spray was actuated and has been operated in the recirculation mode, then the operation in the recirculation mode is required for a duration of

- the 1.5 hours prior to the termination of containment spray to ensure desired pH of the spray solution is reached.
- C. If containment spray is actuated and a primary LOCA is indicated, continuous spray for a minimum duration of 2 hours is required. This should include or be followed by operation in the recirculation mode for at least 1.5 hours.

6.2.2.2.3 Safety Evaluation

- A. The components of the containment spray system are located inside the Category 1 auxiliary and containment buildings, except for the RWST, which is a Category 1 concrete tank. These buildings and this tank are designed to withstand the effects of natural phenomena such as earthquakes, tornadoes, or floods.
- B. Operation of the containment spray system is automatically initiated by a coincident two-out-of-four containment (high-3) pressure signal. Use of this safety-related signal provides a reliable indication of a DBA occurring inside the containment. Operation of the spray system may also be manually initiated from the control room. A detailed description of the actuation system is contained in section 7.3.
- C. Two trains, each containing a spray pump, spray eductor, and spray header, are supplied borated water from the RWST. The spray pump and motor-operated valves in each train are powered from the corresponding train of onsite emergency power. Failure of any active component in one train will not affect the operability of the redundant train. The containment analyses of LOCA and MSLB accidents performed in subsection 6.2.1 assume the availability of only one train of the containment spray system. The results of a failure modes and effects analysis are provided in table 6.2.2-5.
- D. The containment spray system components are qualified to operate in the LOCA and MSLB environments through testing or analysis. Sections 3.9, 3.10, and 3.11 discuss the qualification of mechanical and electrical components.
- E. The analyses described in subsection 6.2.1 show that the spray system in conjunction with the containment coolers is capable of removing sufficient heat energy and subsequent decay heat from the containment atmosphere to ensure the accident peak pressure is below the containment design pressure. The analyses assume the operation of only one train of containment coolers and containment spray.
- F. The containment spray system is designed to Category 1 and Quality Group B requirements as shown in table 3.2.2-1.

As stated in 1.9.26.2, Westinghouse classifies components according to ANSI N 18.2A-1975. The spray additive tanks have been abandoned in place.

The minimum fall path of water droplets is conservatively assumed to be the distance from the lowest spray ring to the operating deck. Heat transfer calculations presented in chapter 15 show that essentially all spray droplets reach thermal equilibrium at containment design temperature and pressure in a distance considerably less than the minimum fall path. For a detailed description of the analytical methods and models used to assess the performance capability of the containment heat removal systems, refer to the containment integrity analysis presented in chapter 15.

6.2.2.2.4 Testing and Inspection

6.2.2.2.4.1 <u>Containment Spray System.</u>

6.2.2.4.1.1 Inspections. The containment spray system is designed to permit periodic determination of proper functioning to demonstrate system readiness as specified in the Technical Specifications.

The pressure containing systems are inspected for leaks from pump seals, valve packings, flange joints, and safety valves during system testing. The components of the system outside the containment are accessible for leaktightness inspection during periodic flow tests.

Examinations/inspections of the pressure retaining piping welds should be performed in accordance with the requirements of applicable codes and standards.

- 6.2.2.2.4.1.2 Preoperational Testing. The objective of preoperational testing is to:
 - A. Demonstrate that the spray nozzles in the containment spray header are clear of obstructions by passing air through the test connections.
 - B. Verify that the proper sequencing of valves and pumps occurs on initiating the containment spray signal, and demonstrate the proper operation of remotely operated valves.
 - C. Verify the operation of the spray pumps; each pump is run at minimum flow, and the flow is directed through the normal path back to the RWST. During this time, the miniflow is measured to verify required flow.
- 6.2.2.2.4.1.3 Operational Testing. The objective of the periodic testing is to:
 - A. Verify that the proper sequencing of valves and pumps occurs on initiation of the containment spray signal, and demonstrate the proper operation of remotely operated valves.
 - B. Verify the operation of the spray pumps; each pump is run at minimum flow, and the flow is directed through normal path back to the RWST.
 - C. Automatic transfer of powered components to the emergency diesel generators can be demonstrated during an integrated system test conducted when the plant is cooled down and the residual heat removal system is in operation.
- 6.2.2.2.4.2 <u>Containment Spray Additive Subsystem</u> (Abandoned in place).

6.2.2.2.5 Instrumentation Application

The status of the containment spray system is displayed in the control room. Using a combination of alarms and monitor lights, the operator is alerted to a maloperation of this equipment both during normal operation and post-accident conditions.

Operation of the containment spray system is demonstrated by monitoring spray water temperature and RWST level. Locally mounted pressure indicators are provided at the spray pumps' suction and discharge to verify pump performance.

The activation signal generating equipment fully meets Institute of Electrical and Electronic Engineers Standard 279 as to operation, diversity, separation of power supplies (diesels, instrument power), etc.; details are discussed in chapter 7.

6.2.2.3 References

- Deleted.
- Deleted.

6.2.3 SECONDARY CONTAINMENT FUNCTIONAL DESIGN

Based on the performance of the fission product removal and control systems discussed in section 6.5 and the acceptable radiological consequences following a loss-of-coolant accident or fuel handling accident inside the containment presented in chapter 15.0, a secondary containment is not required for VEGP.

6.2.4 CONTAINMENT ISOLATION SYSTEM

The containment isolation system consists of the piping, valves, and actuators required to isolate the containment following a loss-of-coolant accident (LOCA), steam line rupture, fuel handling accident inside the containment, small breaks in the reactor coolant system (RCS), or releases of radioactivity from systems within the containment. The design of the containment isolation system satisfies the requirements of TMI Action Plan Task II.E.4.2 as described in the following paragraphs.

6.2.4.1 <u>Design Bases</u>

Protection of the containment isolation system from wind and tornado effects is discussed in section 3.3. Flood design is discussed in section 3.4. Missile protection is discussed in section 3.5. Protection against dynamic effects associated with the postulated rupture of piping is discussed in section 3.6. Environmental design is discussed in section 3.11.

6.2.4.1.1 Safety Design Bases

- A. In the event of a LOCA, the containment isolation system provides isolation of lines penetrating the containment which are not required for operation of the engineered safety features (ESF) systems to minimize the release of radioactive materials to the atmosphere.
- B. Upon failure of a main steam line, the containment isolation system isolates the steam generators as required to prevent excessive cooldown of the RCS or overpressurization of the containment.
- C. To control the release of radioactivity to the outside atmosphere, the containment isolation system isolates the containment atmosphere following a fuel handling accident inside the containment.
 - During refueling operations, manual containment ventilation isolation (CVI) is permitted, along with open personnel and emergency air lock doors. Manual CVI

- capability, using individual valve hand switches, is performed during these refueling operations as described in paragraph 6.2.4.5.
- D. The containment isolation system is designed in accordance with 10 CFR 50, Appendix A, General Design Criterion 54.
- E. Each line which penetrates the containment and which either is a part of the reactor coolant pressure boundary (RCPB) or connects directly to the containment atmosphere or does not meet the requirements for a closed system as defined in item F below, except instrument sensing lines, is provided with containment isolation valves in accordance with 10 CFR 50, Appendix A, General Design Criteria 55 and 56.
- F. Each line which penetrates the containment and is neither part of the RCPB nor connected directly to the atmosphere of the containment and which satisfies the requirements of a closed system is provided a containment isolation valve in accordance with 10 CFR 50, Appendix A, General Design Criterion 57. A closed system is not a part of the RCPB nor connected directly to the atmosphere of the containment and meets the following additional requirements:
 - 1. The system is protected against missiles and the effects of high energy line break.
 - 2. The system is designed to Seismic Category 1 requirements.
 - 3. The system is designed to American Society of Mechanical Engineers Section III, Class 2 requirements.
 - 4. The system is designed to withstand temperatures at least equal to the containment design temperature.
 - 5. The system is designed to withstand the external pressure from the containment structural acceptance test.
 - 6. The system is designed to withstand the design basis accident transient and environment.
- G. The containment pressure transmitters and reactor vessel level instrumentation system (RVLIS) are designed in accordance with Nuclear Regulatory Commission (NRC) Regulatory Guide 1.141. Six containment pressure sensors are provided as sealed systems with bellow seals inside the containment, liquid filled capillaries between the seals, and the sensing element outside containment. RVLIS consists of six level sensors and has bellow seals inside the containment, liquid filled capillaries between the seals, and a secondary isolator seal outside the containment between the containment penetration and the transmitter. These instrument lines are closed systems both inside and outside containment, are designed to withstand the containment pressure and temperature conditions following a loss of coolant accident, and are designed to withstand dynamic effects.
- H. The containment isolation system is designed to remain functional following a safe shutdown earthquake (SSE).

6.2.4.1.2 Power Generation Design Basis

The containment isolation system as a whole has no power generation design basis. Power generation design bases associated with individual components of the containment isolation system are discussed in the section describing the system of which they are an integral part.

6.2.4.2 System Description

6.2.4.2.1 General Description

Each piping system which penetrates the containment is provided with containment isolation features which serve to minimize the release of fission products following a design basis accident. Provisions are made to allow for passage of emergency fluid through the boundary following a postulated accident. Figure 6.2.4-1 provides the arrangement for each piping penetration. NRC Standard Review Plan 6.2.4 and Regulatory Guide 1.141 provide acceptable alternative arrangements to the explicit arrangements given in General Design Criteria 55, 56, and 57. Each penetration is designed so that in the event that a single failure is postulated, the containment integrity is maintained. Table 6.2.4-1 lists each penetration and provides a summary of the containment penetration/isolation valve information.

For those systems which have automatic isolation valves or for which remote-manual isolation is provided, paragraph 6.2.4.5 describes the power supply and associated actuation system. Power-operated (air, motor, electrohydraulic, or solenoid) containment isolation valves have position indication in the control room.

Two modes of valve actuation are considered in table 6.2.4-1. The actuation signal which occurs directly as a result of the event initiating containment isolation is designated as the primary actuation signal. The post-accident valve position is a consequence of the primary actuation signal. If a change in valve position is required at any time following primary actuation, a secondary actuation signal is generated which places the valve in an alternative position. The closure times for automatic containment isolation valves are provided in table 6.2.4-1. Containment isolation valves required to be operable by Technical Specification 3.6.3, Containment Isolation Valves, are demonstrated operable with isolation times as shown in FSAR table 6.2.4-2.

The containment purge system is designed in accordance with Branch Technical Position CSB 6-4 as described in table 9.4.6-4. As described in subsection 9.4.6, the 14-in. minipurge lines may be open during normal plant operation and are provided with isolation valves capable of 5-s closure against the peak calculated containment pressure following a LOCA. The 24-in. purge lines are open only during a cold shutdown condition and are provided with an isolation valve capable of 10-s closure. An analysis of the radiological consequences and the effect on the containment backpressure due to the release of containment atmosphere are discussed in chapter 15 and paragraph 6.2.1.5, respectively.

In the event of a LOCA, the secondary shield wall and other protective features prevent any missiles or high energy line break effects from damaging or degrading the performance capability of the containment isolation system. Sections 3.5 and 3.6 discuss in detail the missiles and pipe break effects, and section 3.8 discusses the internal structures, including the secondary shield wall. The actuators for power-operated containment isolation valves inside the containment are located above the maximum anticipated containment water level. In

addition, lines associated with those penetrations which are considered closed systems inside the containment are protected from the effects of a LOCA.

Provisions are made to ensure that closure of the containment isolation valves is not inhibited by entrapped debris in the valve body. For the majority of the systems, the fluid is demineralized water; thus, process fluid quality does not affect valve operation. For containment minipurge lines, screens are provided in the lines inboard of the isolation valves. For the containment sump lines, including the containment emergency sump, screens are provided to prevent large debris from entering the system.

Other defined bases for containment isolation are provided in NRC Standard Review Plan 6.2.4 and Regulatory Guide 1.141. Conformance with Regulatory Guide 1.141 is provided to the extent specified in this section and in subsections 6.2.5 and 6.2.6. For the emergency core cooling system (ECCS) and containment spray system penetrations, the acceptability of the alternative arrangement relies upon provisions for the detection of possible leakage from these lines outside the containment. Subsection 9.3.3 describes the leak detection provisions that have been made in the plant drainage system. Other provisions, such as containment water level and system flow, temperature, and pressure instrumentation may be used by the operator.

The containment penetrations associated with the secondary side of the steam generators are not subject to General Design Criterion 57. The valves associated with these penetrations do not receive a containment isolation signal and are not credited with effecting containment isolation in the safety analyses. The barriers against fission product release to the environment are the steam generator tubes and the piping associated with the steam generators.

In addition to containment penetration isolation, table 6.2.4-1 also contains systems which are required for post-LOCA mitigation. Since these systems, such as the ECCS, perform additional safety-related functions, they are associated with ESF and are so indicated in table 6.2.4-1.

6.2.4.2.2 Component Description

Codes and standards applicable to the piping and valves associated with containment isolation are listed in table 3.2.2-1. Containment penetrations are classified as Quality Group B and Seismic Category 1.

Section 3.11 provides the post-LOCA environment that is used to qualify the operability of power-operated isolation valves located inside the containment.

The containment penetrations are designed to meet the stress requirements of NRC Branch Technical Position MEB 3-1 and the classification and inspection requirements of NRC Branch Technical Position APCSB 3-1, as described in section 3.6. Section 3.8 discusses the interface between the piping system and the containment liner.

6.2.4.2.3 System Operation

During normal operation, many penetrations are not isolated. Lines which are not required for the passage of emergency fluids are automatically isolated upon receipt of isolation signals, as discussed in paragraphs 6.2.4.3 and 6.2.4.5 and chapter 7. Essential lines which penetrate the containment are closed loops within the containment or provide flow paths into or out of the RCS and can be isolated by remote-manual operation when dictated by the emergency system functional requirements. Lines not in use during power operation are normally closed and remain closed under administrative control during reactor operation.

Upon detection of abnormal radioactivity levels indicative of a fuel handling accident during refueling or other release, the isolation valves in the containment purge system are closed to minimize any fission product release to the environment.

6.2.4.3 Design Evaluation

Safety evaluations are lettered to correspond to the safety design bases.

A. Containment isolation signals automatically isolate process lines which are nonessential as identified in table 6.2.4-1. Nonessential lines are those lines which are not required to mitigate or limit an accident and which, if required at all, would be required for long term recovery only; e.g., days or weeks following an accident.

Lines which are required to mitigate an accident or which, if unavailable, could increase the magnitude of the event are designated as essential lines. Table 6.2.4-1 identifies the associated line as essential or nonessential and shows the automatic isolation signal for each penetration, if applicable.

The containment isolation system utilizes diversity in the parameters sensed for the initiation of containment isolation. The two redundant train-oriented containment isolation phase A signals (CIA-A, CIA-B) are initiated on receipt of any of the following signals:

- 1. Any signal initiating a safety injection:
 - Manual safety injection actuation.
 - High containment pressure (high-1).
 - Low steam line pressure.
 - Low pressurizer pressure.
- 2. Manual containment isolation actuation.
- B. Upon failure of a main steam line, the steam generators are isolated to prevent excessive cooldown of the RCS or overpressurization of the containment.

The two redundant train-oriented steam line isolation signals (SLI-A, SLI-B) are initiated upon receipt of any of the following signals:

- 1. High steam line pressure rate.
- 2. Low steam line pressure.
- 3. Containment high-2 pressure.
- Manual actuation.

For main steam line breaks resulting in a high steam line pressure rate or containment high-2 pressure signal, only the main steam line isolation valves (MSIVs) and MSIV bypass valves are shut to prevent excessive cooldown of the RCS. When the main steam line break causes a low steam line pressure signal, a safety injection signal (followed by containment isolation) is generated as well as the steam line isolation signal.

The main steam line isolation valves, MSIV bypass valves, and piping are designed to prevent uncontrolled blowdown from more than one steam generator. For the events analyzed, the main steam line isolation valves shut

fully within 7 s after receipt of the SLI signal. The MSIV bypass valves will shut fully within 5 s after receipt of SLI signal. The blowdown rate is restricted by steam flow restrictors located within the steam generator outlet steam nozzles in each blowdown path. For main steam line breaks upstream of an isolation valve, uncontrolled blowdown from more than one steam generator is prevented by the isolation valves in the unaffected steam lines and by the isolation valve in the affected line. For main steam line breaks downstream of an isolation valve, blowdown from more than one steam generator is prevented by the main steam isolation valves on each main steam line.

Failure of any one of the above components relied upon to prevent uncontrolled blowdown of more than one steam generator will not permit a second steam generator blowdown to occur. Piping restraints and pipe whip barriers between the main steam lines prevent a rupture in one line from causing a blowdown from more than one steam generator. No single active component failure will result in the failure of more than one main steam isolation valve to operate. Redundant main steam isolation signals, described in section 7.3, are fed to redundant parallel activation cylinder vent valves and redundant series actuation cylinder air supply valves to ensure isolation valve closure in the event of a single isolation signal failure.

The effect on the RCS after a steam line break resulting in single steam generator blowdown and the offsite radiation exposure after a steam line break outside containment are discussed in detail in chapter 15. The containment pressure transient following a main steam line break inside containment is discussed in section 6.2.

- C. The containment purge system is automatically isolated following an abnormal release of radioactivity in the containment by either of two redundant trainoriented containment ventilation isolation signals (CVI-A, CVI-B) generated upon receipt of any of the following:
 - 1. Any signal resulting in a safety injection.
 - 2. Containment high area radiation.
 - 3. Containment high radioactive air particulate.
 - 4. Containment high radioactive gas.
 - 5. Containment high iodine concentration.
 - 6. Manual actuation of either containment spray or containment isolation phase A.

The preaccess purge supply and exhaust valves in the 24-in. lines, which are only open in the cold shutdown condition, are designed to shut in less than 10 s. The minipurge line isolation valves, which may be open during normal operation, shut in less than 5 s.

D. The containment isolation system is designed in accordance with 10 CFR 50, Appendix A, General Design Criterion 54. Leakage detection capabilities and the leakage detection test program are discussed in subsection 6.2.6. Valve operability tests are also discussed in subsection 3.9.6. Redundancy of valves and reliability of the isolation system are ensured by conformance with the other safety design bases stated in section 6.2. Redundancy and reliability of the actuation system are covered in section 7.3.

The use of motor-operated valves which fail as is upon loss of actuating power in lines penetrating the containment is based upon the consideration of what valve position ensures the greatest plant safety. Furthermore, each of these valves that fails as is provided with redundant backup valves to ensure that no single failure will prevent the system as a whole from performing its isolation function; e.g., a check valve inside the containment and motor-operated valve outside the containment or two motor-operated valves in series, each powered from a separate ESF bus.

- E. Lines which penetrate the containment and which either are part of the RCPB, connect directly to the containment atmosphere, or do not meet the requirements for a closed system, except instrument sensing lines, are provided with one of the following valve arrangements conforming to the requirements of 10 CFR 50, Appendix A, General Design Criteria 55 and 56, as follows:
 - 1. One locked closed isolation valve inside and one locked closed isolation valve outside containment.
 - 2. One automatic isolation valve inside and one locked closed isolation valve outside containment.
 - 3. One locked closed isolation valve inside and one automatic isolation valve outside containment. (A simple check valve is not used as the automatic isolation valve outside containment.)
 - 4. One automatic isolation valve inside and one automatic isolation valve outside containment. (A simple check valve is not used as the automatic isolation valve outside containment.)

Isolation valves outside containment are located as close to the containment as practical, and upon loss of actuating power, air-operated automatic isolation valves fail closed.

- F. Each line which penetrates the containment and is neither part of the RCPB nor connected directly to the containment atmosphere and satisfies the requirements of a closed system has at least one containment isolation valve which is either automatic, locked closed, or capable of remote-manual operation. The valve is outside the containment and located as close to the containment as practical. A simple check valve is not used as the automatic isolation valve. This design is in compliance with 10 CFR 50, Appendix A, General Design Criterion 57.
- G. Instrument lines penetrating the containment and the containment pressure instrument lines are designed in accordance with NRC Regulatory Guide 1.141.
- H. The containment isolation system is designed in accordance with Seismic Category 1 requirements as specified in section 3.2. The components (and supporting structures) of any system, equipment, or structure which is non-Seismic Category 1 and whose collapse could result in loss of a required function of the containment isolation system through either impact or flooding are analytically checked to determine that they will not collapse when subjected to seismic loading resulting from an SSE.

Air-operated isolation valves fail in the shut position upon loss of air if they are not required to operate after a design basis accident. Containment isolation system valves required to be operated after a design basis accident are powered by the Class 1E electric power system.

6.2.4.4 <u>Tests and Inspections</u>

Preoperational testing is described in chapter 14. The containment isolation system is testable through the operational sequence that is postulated to take place following an accident, including operation of applicable portions of the protection system and the transfer between normal and standby power sources.

The piping and valves associated with the containment penetration are designed and located to permit preservice and inservice inspection in accordance with ASME Section XI, as discussed in section 6.6.

Each line penetrating the containment is provided with testing features to allow containment leak rate tests in accordance with 10 CFR 50, Appendix J, as discussed in subsection 6.2.6.

6.2.4.5 Instrumentation Application

The generation of CIA or CVI signals which automatically isolate the appropriate containment isolation valves is described in section 7.3.

For those valves for which automatic closure is not desired, based on the system safety function, remote-manual operation is available from the control room.

Containment isolation valves which are equipped with power operators and which are automatically actuated may also be controlled individually by positioning hand switches in the control room. Also, in the case of certain valves with actuators, a manual override of an automatic isolation signal is installed to permit manual control of the associated valve. The override control function can be performed only subsequent to resetting of the actuation signal; that is, deliberate manual action is required to change the position of containment isolation valves in addition to resetting the original actuation signal. The design does not allow ganged reopening of the containment isolation valves. Reopening of the isolation valves must be performed on a valve-by-valve basis, or on a line-by-line basis. Safety injection signals take precedence over manual overrides of other isolation signals, for example, a safety injection signal causes isolation valve closure even though the high radiation signal is being overridden by the operator. Overrides are input to the system status monitoring panel, described in subsection 7.5.5. Containment isolation valves with power operators are provided with open/closed indication, which is displayed in the control room. The valve mechanism also provides a local, mechanical indication of valve position.

In mode 6 during core alterations and movement of irradiated fuel assemblies inside containment, automatic or system-level manual initiation CVI capability no longer applies. During these refueling operations, manual containment ventilation isolation is permitted, along with open personnel and emergency air lock doors. Manual CVI capability, using individual valve hand switches, is performed during these refueling operations. The following conditions apply:

- One personnel air lock door and one emergency air lock door must be operable, and
- At least 23 feet of water is maintained above the reactor vessel flange, and
- A designated individual is available to close the doors. The emergency air lock will
 not normally be open during core alterations or fuel movement inside containment.
 Therefore, in the event the emergency air lock is open at the same time the
 personnel air lock is open, a separate individual shall be responsible for closing the

emergency air lock (within 15 minutes) in addition to the individual designated to close the personnel air lock.

All power supplies and control functions necessary for containment isolation are Class 1E, as described in chapters 7 and 8.

6.2.5 COMBUSTIBLE GAS CONTROL IN CONTAINMENT

Technical Specifications (TS) Amendment Number 134/113 eliminated the requirements regarding containment hydrogen recombiners and relaxed the requirements for hydrogen monitors. The hydrogen recombiners and monitors have been deleted from the TS. The hydrogen monitors are included in the post accident monitoring instrument program.

Following a loss-of-coolant accident (LOCA), hydrogen may be produced inside the reactor containment by radiolysis of the core and sump solutions, by corrosion of aluminum and zinc, by reaction of the Zircaloy fuel cladding with water, and by release of the hydrogen dissolved in the reactor coolant and contained in the pressurizer vapor space. To ensure that the containment hydrogen concentration is maintained at a level low enough to preclude endangering containment integrity, a combustible gas control system is provided. This subsection describes the systems that are provided in accordance with General Design Criterion 41 to control the buildup of hydrogen within the containment.

Five mechanisms for monitoring and controlling hydrogen inside the containment are considered in the VEGP design:

- Hydrogen recombiners.
- Post-LOCA containment hydrogen purge.
- Post-LOCA cavity hydrogen purge.
- Containment hydrogen monitoring.
- Containment hydrogen mixing.

6.2.5.1 Design Bases

6.2.5.1.1 Electric Hydrogen Recombiners

The following design bases apply to the electric hydrogen recombiners:

- A. The recombiners are designed to sustain all normal and accident loads including safe shutdown earthquake and pressure transients from a design basis LOCA.
- B. The recombiners are designed for a lifetime of 40 years, consistent with that of the plant.^a

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^a The operating licenses for both VEGP units have been renewed and the original licensed operating terms have been extended by 20 years, resulting in a plant operating life of 60 years. In accordance with 10 CFR Part 54, appropriate aging management programs and activities have been initiated to manage

- C. All materials used in the recombiners are selected to be compatible with the environmental conditions inside the reactor containment during normal operation or during accident conditions.
- D. Process capacity is such that the containment hydrogen concentration will not exceed 4 volume% based on the Nuclear Regulatory Commission (NRC) TID release model as indicated in Regulatory Guide 1.7.
- E. Two redundant electric hydrogen recombiners are provided to meet the single-failure criterion.

6.2.5.1.2 Post-LOCA Containment Hydrogen Purge System

- A. The containment post-LOCA purge exhaust system constitutes a 100% single train backup to the hydrogen recombiner. It is designed as Seismic Category 1 except for ductwork inside the containment upstream of the containment isolation valve and ductwork at the filter inlet.
- B. The post-LOCA containment pressure provides the motive force to purge the containment.
- C. This system may be used post-LOCA to maintain hydrogen levels in the containment below 4 volume% in conjunction with a portable air compressor through the Seismic Category 1 portion of the service air piping.
- D. The redundant motor-operated isolation valves located inside the containment are in parallel and powered from the 480-V, Class 1E buses. All other system components are powered from the normal ac bus. The outside isolation valve is locked closed and manually controlled by the operator. The isolation valves inside the containment have position indication in the control room and are normally closed.
- E. The system, including the isolation valves and the filtration unit, is designed to be manually started for operation.

6.2.5.1.3 Post-LOCA Cavity Hydrogen Purge System

- A. The post-LOCA cavity purge system prevents hydrogen pocketing in the reactor cavity after a LOCA by supplying air to the reactor cavity for dilution.
- B. The post-LOCA cavity purge system is capable of accomplishing its function with the single failure of an active component.
- C. The post-LOCA cavity purge system is capable of performing its function with Class 1E power, each redundant train being connected to separate Class 1E safety buses.
- D. The system is designed to automatically start upon receipt of a safety injection signal.
- E. The system is designed to meet Seismic Category 1 requirements.

the detrimental effects of aging to maintain functionality during the period of extended operation (see chapter 19).

6.2.5.1.4 Containment Hydrogen Monitoring System

- A. The hydrogen monitoring system is designed as a Class 1E, Seismic Category 1 system. It is designed to retain its integrity and operability following a design basis accident (DBA).
- B. All materials and equipment required by this system are selected to be compatible with the environmental conditions anticipated during accident operation and are suitable for a lifetime consistent with that of the plant^a.
- C. The system samples containment air, providing the means to measure the containment hydrogen concentration and to alert the operator in the event that a high hydrogen concentration is detected, in accordance with the requirements of Regulatory Guide 1.7.
- D. The hydrogen monitoring system consists of two identical units that are completely independent of each other and are powered from independent Class 1E power sources. Assuming a single failure and compensatory operator actions from the control room, capability is available to monitor the hydrogen concentration in the containment.
- E. Proper shielding and other provisions are incorporated into the design to ensure that personnel exposure does not exceed the limits of General Design Criterion 19 and that the required radiological analysis can be performed on the containment air sample.

6.2.5.1.5 Containment Hydrogen Mixing

The following design bases apply to mechanisms or systems for mixing of hydrogen-bearing gases inside the reactor containment:

- A. Local hydrogen concentrations inside the reactor containment shall be maintained at less than 4 volume%.
- B. Any active systems required for mixing containment air should meet the same redundancy, environmental, seismic, and quality requirements as the hydrogen recombiner system as described in paragraph 6.2.5.1.1.

6.2.5.2 System Design

6.2.5.2.1 Electric Hydrogen Recombiners

The applicable codes and standards used in the design of the electric hydrogen recombiner are listed in table 6.2.5-1, and a typical electrical recombiner is shown in figure 6.2.5-1.

Each recombiner system consists of a control panel located in the control building, a power supply cabinet located on level B of the control building, and a recombiner located above the

^a The operating licenses for both VEGP units have been renewed and the original licensed operating terms have been extended by 20 years, resulting in a plant operating life of 60 years. In accordance with 10 CFR Part 54, appropriate aging management programs and activities have been initiated to manage the detrimental effects of aging to maintain functionality during the period of extended operation (see chapter 19).

operating deck at el 261 ft in the containment. There are no moving parts or controls inside the containment. Heating air within the unit causes airflow by natural convection. The recombiner is a completely passive device.

To regulate the power supply to the recombiner, the power supply cabinet located in the control building contains an isolation transformer and a controller. This equipment will not be exposed to the post-LOCA environment. The controls for the power supply are located in the control building beside the power supply panel and are manually actuated.

Each hydrogen recombiner consists of the following components:

- A. A preheater section, consisting of a shroud placed around the central heaters to take advantage of heat conduction through the central walls, for preheating incoming air.
- B. An orifice plate to regulate the rate of airflow through the unit.
- C. A heater section, consisting of four banks of metal-sheathed electric resistance heaters, to heat the air flowing through it to hydrogen-oxygen recombination temperatures.
- D. An exhaust chamber which mixes and dilutes the hot effluent with containment air to lower the temperature of the discharge stream.
- E. An outer enclosure to protect the unit from impingement by containment spray.

The hydrogen recombiner has no need for external services except electrical power.

The containment atmosphere is heated within the recombiner in a vertical duct, causing it to rise by natural convection. As it rises, replacement air is drawn through intake louvers downward through a preheater section which will temper the air and lower its relative humidity. The preheated air then flows through an orifice plate, sized to maintain a 100-sf³/min flowrate, to the heater section. The airflow is heated to a temperature above 1150°F, the reaction temperature for the hydrogen-oxygen reaction. Any free hydrogen present reacts with atmospheric oxygen to form water vapor. After passing through the heater section, the flow enters a mixing section, which is a louvered chamber where the hot gases are mixed and cooled with containment atmosphere before the gases are discharged directly into the containment. The air discharge louvers are located on three sides of the recombiner. To avoid short-circuiting previously processed air, no discharge louvers are located on the intake side of the recombiner.

Tests have verified that the hydrogen-oxygen recombination is not a catalytic surface effect associated with the heaters (paragraph 6.2.5.4) but occurs due to the increased temperature of the process gases. As the phenomenon is not a catalytic effect, saturation of the unit cannot occur.

Two recombiners are provided to meet the requirements for redundancy and independence. Each recombiner is powered from a separate safeguard bus and is provided with a separate power panel and control panel. This system and the other safety-related subsystems are not interdependent.

The unit is manufactured of corrosion-resistant, high-temperature material. The electric hydrogen recombiner uses commercial-type electric resistance heaters sheathed with Incoloy-800, which is an excellent corrosion-resistant material for this service. The recombiner heaters operate at significantly lower power densities than similar heaters used in commercial practice.

The recombiner is operated manually from a control panel located in the control building. The recombiner, power supply panel, and control panel are shown schematically in figure 6.2.5-2. The power panel for the recombiner contains an isolation transformer and a controller to

regulate power into the recombiner. This equipment is not exposed to the post-LOCA containment environment.

To control the recombination process, the correct power input to bring the recombiner above the threshold temperature for recombination is set on the controller. The correct power required for recombination depends upon containment atmosphere conditions and is determined when recombiner operation is required. A thermocouple readout instrument is also provided in the control panel to monitor temperatures in the recombiner.

6.2.5.3 System Design

6.2.5.3.1 Containment Hydrogen Purge System.

The post-LOCA containment hydrogen purge system is provided as a backup means of controlling hydrogen inside the containment. It provides a means of purging the hydrogen from the containment and is intended as a backup to the hydrogen recombiner system. The exhaust filters and exhaust duct to plant vent are designed as Seismic Category 1.

The system consists of an exhaust penetration line and a filtered exhaust system; it is shown in drawings 1X4DB213-1 and 1X4DB213-2. Design data for principal system components are presented in table 6.2.5-3. The containment isolation valves and interconnecting piping are Seismic Category 1; all other portions of the system are Seismic Category 2.

The purge exhaust filter unit includes in the direction of airflow:

- A demister
- Electrical heating coil.
- High-efficiency particulate air (HEPA) prefilter.
- A 4-in, charcoal adsorber.
- A HEPA afterfilter.

The hydrogen purge exhaust filter unit is located in the equipment building, and the hydrogen purge intake point is located in the containment dome. The ductwork is fastened to the inside of the containment and routed through a containment penetration. The flowrate through the filter unit is 500 ft³/min.

The system is actuated manually. The operator unlocks and opens the manual isolation valve located outside the containment. From the control room, the operator opens the remote-manual isolation valves located inside the containment. The outward purge flow is due to the pressure differential existing between the containment atmosphere and the environment. Periodically, the operator will dilute the containment atmosphere by charging air into the containment via the instrument and service air system.

6.2.5.3.2 Post-LOCA Cavity Purge System

The post-LOCA cavity purge system is designed to prevent hydrogen pocketing in the reactor cavity following a LOCA by supplying air to the reactor cavity for dilution of the hydrogen

released in the cavity area. The system fans take a suction on the atmosphere within the generator compartments and discharge the air into the cavity above (el 193 ft 2 in.) and below (169 ft 9 in.) the reactor vessel nozzles. The air flows out of the cavity along the reactor vessel nozzles or through the ventilation openings surrounding the seal ring.

The system consists of two 100%-capacity fans. Each fan is powered from an independent Class 1E power supply. The fans automatically start on a safety injection signal. There is an independent discharge pipe which routes the dilution air from the fans to the cavity. A failure modes and effects analysis is provided in table 6.2.5-2. Distribution of the air in the cavity area is accomplished via common discharge headers. All portions of the system are designed to Seismic Category 1 requirements.

Design data for principal components are provided in table 6.2.5-4. The system is schematically shown in drawing 1X4DB214-2.

6.2.5.3.3 Containment Hydrogen Monitoring System

Each redundant hydrogen monitoring train in the hydrogen monitoring system consists of a hydrogen analyzer and two associated sample lines with solenoid-operated isolation valves inside and outside the containment. These sampling lines are designed to be free of water traps (runs where liquid could accumulate) and are equipped with sufficient heat tracing to prevent condensation from the sample being supplied to the analyzers.

After the sample has been analyzed, it is returned to the containment. The analyzers are located in accessible areas outside the containment. The hydrogen monitoring subsystem piping is in accordance with the criteria of Regulatory Guide 1.26, Quality Group B. Solenoid-operated isolation valves are arranged to obtain samples from two locations within the containment for each train. The operator may select either of these sampling points from the main control room.

The operation of the hydrogen gas analyzer is based on the measurement of thermal conductivity of the gaseous containment atmosphere sample. The thermal conductivity of the gas mixture changes in proportion to the changes in the concentration of the individual gas constituents of the mixture. The thermal conductivity of hydrogen is far greater (approximately seven times the thermal conductivity of air) than any other gases or vapors expected to be present. This operation of the hydrogen monitoring system is not limited due to radiation, moisture, or temperature expected at the equipment location. The monitors are designed to function under design pressure conditions of -2 to 60 psig.

The containment hydrogen monitors are aligned for operation within 60 minutes after initiating safety injection following a LOCA. Accurate indication of hydrogen concentration is available within 30 min of initiating flow through the monitors. This is accomplished by operating the monitors in standby during normal plant operation. Therefore, indication of containment hydrogen concentration is available to the operators within 90 minutes of initiating safety injection following a LOCA.

The range of the monitors is 0 to 10 volume% with an accuracy of ±5.0% of scale.

The output signal of the hydrogen monitors is indicated and alarmed locally as well as indicated, recorded, and alarmed in the control room. In addition to the high hydrogen alarm, a common malfunction alarm is located in the control room to indicate loss of power, low gas pressure, low analyzer chamber temperature, analyzer cell failure, or high hydrogen concentration.

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Design data for principal system components are presented in table 6.2.5-5. The system is schematically shown in drawings 1X4DB213-1 and 1X4DB213-2.

The hydrogen monitoring system meets the requirements of TMI Action Plan Task II.F.1 with the clarification that accurate indication of containment hydrogen concentration is available to the operators within 90 minutes of initiating safety injection following a LOCA.

6.2.5.3.4 Containment Hydrogen Mixing

Hydrogen mixing is facilitated by the containment fan coolers, which take suction from above the operating deck and discharge to the lower levels of the containment. Functional descriptions of the containment coolers are provided in subsections 6.2.2 and 9.4.6. A flow diagram for the containment coolers is provided in drawings 1X4DB251-1, 1X4DB252, and 1X4DB253-1.

In addition, the post-LOCA cavity purge system described in paragraph 6.2.5.2.3 is available for hydrogen mixing.

6.2.5.4 Design Evaluation

6.2.5.4.1 Hydrogen Production and Accumulation

6.2.5.4.1.1 Zirconium-Water Reaction. A major source of hydrogen immediately following a LOCA is caused by the reaction of the Zircaloy fuel cladding with water. The extent of the zirconium-water reaction depends upon the effectiveness of the emergency core cooling systems (ECCS). An evaluation of the VEGP ECCS shows the zirconium-water reaction to be less than 0.3%.

Zirconium reacts with steam according to the following equation:

$$Zr + 2 H2O \rightarrow Zr O2 + 2 H2 + Heat$$

The hydrogen produced is calculated as follows:

$$\frac{2 \text{ lb} - \text{mole H}_2/\text{lb} - \text{mole Zr}}{91.22 \text{ lb Zr/lb} - \text{mole Zr}} = \frac{0.022 \text{ lb} - \text{mole H}_2}{\text{lb Zr}}$$

The NRC model suggested in Regulatory Guide 1.7 and Standard Review Plan 6.2.5 conservatively assumes a 1.5% zirc-water reaction (five times the maximum amount calculated in the ECCS evaluations).⁽¹⁾ There are approximately 45,914 lb of zirconium metal in the reactor core. The hydrogen produced by the reaction of 689 lb of zirconium is 15.15 lb-moles. This hydrogen is assumed to be immediately released to the containment atmosphere.

6.2.5.4.1.2 <u>Radiolysis Core and Sump Solutions</u>. Water radiolysis is a complex process involving reactions of numerous intermediates. However, the overall radiolytic process may be described by the equation:

$$H_2O \leftrightarrow H_2 + \frac{1}{2} O_2$$

An extensive program was conducted by Westinghouse to investigate the radiolytic decomposition of the core cooling solution following the DBA. During the investigation it

became apparent that post-accident conditions in the containment create two distinct radiolytic environments. One environment exists inside the reactor vessel, where radiolysis can occur when energy emitted by decaying fission products in the fuel is absorbed by the solution pumped through the reactor to cool the core. The other environment exists outside the reactor vessel, in the containment sump solution, where radiolysis can also occur when decay energy emitted by dissolved fission products is absorbed by the sump solution. The two basic differences between the core environment and the sump environment that affect the rate of hydrogen production are the rate of energy absorption and the type of flow regime. The results of these investigations are discussed in reference 1.

The rate of hydrogen production by radiolysis depends upon the rate of energy absorption by the solution. A detailed analysis of energy deposition in the reactor core where decaying fission products are retained in the fuel shows that beta radiation (which represents roughly 50% of the total decay energy) is emitted at an energy level too low to permit its penetration of the fuel and cladding. As a result, roughly 50% of the total decay energy emitted by fission products in the fuel is absorbed by the fuel and cladding and therefore does not contribute significantly to the rate of energy absorption by the water. Furthermore, approximately 7% of the gamma energy is absorbed by the core solution; the rest is absorbed by the fuel, cladding, or other core components.

In the containment sump, where fission products are assumed to be dissolved in the sump solution, energy is emitted directly to the solution. Since the depth of the sump is relatively large compared to the penetrating capability of even gamma energy, effectively 100% of the decay energy of the fission products dissolved in solution is absorbed by the solution. The other significant difference between the core and sump environment is the type of flow regime to which the products of radiolysis are exposed.

Radiolytic decomposition of water is a reversible reaction. In the core, where the products of radiolysis are continuously flushed away by the circulation of cooling solutions, there is little chance for hydrogen and oxygen to accumulate. Consequently, recombination of hydrogen and oxygen is assumed not to occur because significant quantities of the two reactants are not available. The sump, however, is a relatively deep and static environment, where the products of radiolysis are removed by molecular diffusion. Experimental tests simulating sump conditions demonstrate that there is significant reverse reaction in the sump. Hence, there is an apparent reduction in the quantity of hydrogen produced per unit energy absorbed.

The results of Westinghouse and Oak Ridge National Laboratory studies indicate maximum hydrogen yields of 0.44 molecules per 100 eV for core radiolysis and 0.3 molecules per 100 eV for sump radiolysis. The results of these studies are published in references 2, 3, and 4. This analysis, based on the conservative recommendations of Regulatory Guide 1.7 and Standard Review Plan 6.2.5, assumes a hydrogen yield of 0.5 molecules per 100 eV of energy absorbed for both core and sump radiolysis.

The rate of hydrogen gas production from radiolysis has been evaluated using the methodology presented in Appendix A to Standard Review Plan 6.2.5, and the calculational assumptions detailed in Regulatory Guide 1.7. Table 6.2.5-6 provides a summary of the assumptions made in the analysis.

6.2.5.4.1.3 <u>Corrosion of Metals and Paints in Containment</u>. Following a LOCA, hydrogen may be produced inside the containment by corrosion of aluminum and zinc.

Extensive corrosion testing has been conducted to determine the behavior and compatibility of various materials with alkaline borate solution. (7)(8)(9) Metals tested included Zircaloy, Inconel,

aluminum alloys, cupronickel alloys, carbon steel, galvanized carbon steel, and copper. The tests showed that only aluminum and zinc will corrode at a rate that will significantly add to the hydrogen accumulation in the containment atmosphere.

Aluminum is found in the containment as aluminum metal components. Zinc will be in the form of either galvanized steel, zinc metal, or zinc-based paint.

Aluminum corrosion may be described by the overall reaction:

$$2 AI + 3 H2O \rightarrow AI2O3 + 3 H2$$

Three moles of hydrogen gas are produced for every two moles of aluminum that is oxidized. Approximately 0.0556 lb-moles of hydrogen gas are produced for each pound of aluminum corroded.

The corrosion of zinc may be described by the overall reaction:

$$Zn + H_2O \rightarrow Zn(OH)_2 + H_2$$

One mole of hydrogen gas is produced for each mole of zinc that is oxidized. Approximately 0.0153 lb-moles of hydrogen gas are produced for each pound of zinc corroded.

The time-temperature cycle (table 6.2.5-9) considered in the calculation of aluminum and zinc corrosion is a representation of the postulated post-accident containment temperature transient. The corrosion rates at the various temperatures are shown in table 6.2.5-10. With these corrosion rates and the baseline aluminum and zinc inventory given in table 6.2.5-6, the contribution of aluminum and zinc corrosion to the hydrogen accumulation in the containment following the DBA was calculated. No credit was taken for the protective shielding effects of insulation or enclosures; i.e., complete and continuous immersion in spray was assumed.

Calculations based on Regulatory Guide 1.7 are performed by increasing the aluminum corrosion rate during the final interval of the post-accident containment temperature transient (table 6.2.5-9) to 200 mils/year. The aluminum and zinc corrosion rates earlier in the accident sequence are shown in table 6.2.5-10.

In order to determine an allowable margin of increase for the zinc and aluminum inventories, the hydrogen accumulation in the containment following a LOCA was calculated for different cases of varying zinc and aluminum inventories. Three cases were evaluated. They included the case where no recombiners were in operation, the case where one recombiner is in operation starting at day 2, and the case where one recombiner is in operation at 3.5 v/o of hydrogen. Based on the zinc and aluminum inventories provided by table 6.2.5-6, it was determined that the bounding case is 3 times the baseline zinc inventory or 3 times the baseline aluminum inventory or any combination thereof which satisfies the following equation:

$$\frac{N_{al}}{O_{al}} + \frac{N_{zn} + (N_{zbp} - O_{zbp})}{O_{zn}} \le 3.8$$

where:

N_{al} = new total surface area of aluminum (ft²) O_{al} = baseline surface area of aluminum

 $O_{al} = 3607 \text{ ft}^2 \text{ (table 6.2.5-6)}$

N_{zn} = new total surface area of zinc (ft²) O_{zn} = baseline surface area of zinc

 O_{zn} = 182,053 ft² (table 6.2.5-6)

N_{zbp} = new total surface area of zinc-based paint (ft²)

O_{zbp} = baseline surface area of zinc-based paint

 $O_{zbp} = 680,722 \text{ ft}^2 \text{ (table 6.2.5-6)}$

Future additions of aluminum and zinc beyond those quantities identified in table 6.2.5-6 are documented and used in this equation to ensure that the limits of hydrogen generation are not exceeded.

The above equation is valid if at least one recombiner is turned on at a hydrogen concentration of 3.03 v/o (2 days), or less, following a LOCA. It is dependent on surface area only and not dependent on the total mass of zinc and aluminum in containment.

Although 4.0 v/o is the acceptance limit, the maximum H_2 v/o should not exceed 3.3 v/o to account for instrument inaccuracies.

6.2.5.4.1.4 <u>Hydrogen in the Primary Coolant</u>. During normal operation of the plant, hydrogen is dissolved in the reactor coolant and is also contained in the pressurizer vapor space. Following a LOCA, this hydrogen is assumed to be immediately released to the containment atmosphere. The maximum equilibrium quantity of hydrogen from this source is 1600 sf³.

The pressurizer vapor space hydrogen is based on the following:

- A. Reactor coolant hydrogen concentration of 50 cm³ (STP)/kg of coolant.
- B. Normal pressurizer heaters turned on 50% of the time and all of the heat going to the boiling water.
- C. Bypass spray rate of 1 gal/min.
- D. Normal liquid level in pressurizer (60%).
- E. Pressurizer relief valves closed.

6.2.5.4.1.5 <u>Hydrogen Mixing</u>. Experiments (references 10 through 16) demonstrate that for the period of high hydrogen evolution during and following blowdown, bulk turbulence and natural convective transport will be available to distribute and diffuse hydrogen throughout the containment.

Following this period, long-term mixing within and between the lower volumes of the containment and the region above the operating deck will be provided by the containment sprays (if operating) and the emergency containment coolers. The reactor vessel head vent system will provide for the release of any concentrated hydrogen from the primary loop at a controlled rate and in a location so as to allow complete dispersion due to the natural diffusion tendencies of hydrogen and by augmented means such as the containment cooler discharge, which is ducted so as to maintain hydrogen concentration equilibrium between the upper and lower containment regions through forced convection.

6.2.5.4.1.6 <u>Conclusions</u>. Figure 6.2.5-5 shows bulk containment volume% hydrogen versus time following a LOCA. The results from the post-LOCA hydrogen generation calculation have determined that for the three cases of varying aluminum/zinc inventory with a single 95% efficient, 100-sf³/min recombiner starting operation on the second day following a LOCA or when the containment hydrogen concentration reaches 3.03 v/o, the hydrogen concentration

peaks below 3.5 v/o and is maintained well below 4.0 v/o, thus showing ample margin in the hydrogen control system.

The integrated hydrogen production for each source is shown in figure 6.2.5-7.

6.2.5.5 Tests and Inspections

6.2.5.5.1 Electric Hydrogen Recombiners

The electric hydrogen recombiners underwent extensive testing in the Westinghouse development program. These tests encompassed the initial analytical studies, laboratory proof-of-principal tests, and full-scale prototype testing. The full-scale prototype tests included the effects of:

- Varying hydrogen concentrations.
- Alkaline spray atmosphere.
- Steam effects.
- Convection currents.
- Seismic effects.

A detailed discussion of these tests is provided in references 17 through 24.

6.2.5.5.2 Post-LOCA Containment Hydrogen Purge System

Safety-related equipment is qualified by the vendor to meet the codes and standards required by the system classification. Functional testing is performed after installation but prior to plant startup to verify the system performance capability. Periodic testing of the system components will be performed in accordance with plant procedures.

6.2.5.5.3 Post-LOCA Cavity Hydrogen Purge System

Safety-related equipment is qualified by the vendor to meet the codes and standards required by the system classification. Functional testing is performed after installation but prior to plant startup to verify the system performance capability.

6.2.5.5.4 Post-Accident Hydrogen Monitoring System

Equipment for this system is vendor qualified to meet the codes and standards required by the system classification. Functional and preoperational testing is performed after installation and prior to plant startup to verify the system performance capability. Periodic testing of the system and the isolation and sample selector valves will be performed in accordance with plant procedures.

6.2.5.6 <u>Instrumentation Requirements</u>

6.2.5.6.1 Electric Hydrogen Recombiner

The recombiners do not require any instrumentation inside the containment for proper operation after a LOCA. The recombiners are started manually after a LOCA. The sampling system is used to obtain containment atmosphere samples that indicate when the recombiners or the venting system should be actuated. Control measures can be initiated when the hydrogen concentration reaches 3 volume%.

6.2.5.6.2 Post-LOCA Containment Hydrogen Purge System

Instrumentation and controls for this system are located outside of the containment in the equipment building or in the control room. Control switches and status indication for the fans, isolation valves, and control valves are provided in the control room. Indication monitoring the operation of the filter exhaust unit is also provided in the control room.

6.2.5.6.3 Post-LOCA Cavity Hydrogen Purge System

Control switches for manual operation of the fans and low-flow alarms for each discharge line are provided on the HVAC panel in the control room.

6.2.5.6.4 Containment Hydrogen Monitoring System

The control switches for the sample selector valves and containment isolation valves are located on the process control panel in the control room. Operation of the hydrogen analyzers is controlled remotely from the main control board. Hydrogen concentration is both indicated and recorded on the main control board.

6.2.5.7 Materials

The materials of construction for the hydrogen control systems are selected for their compatibility with the post-LOCA environment.

The major structural components of the hydrogen recombiners are manufactured from 300-Series stainless steel. Incoloy-800 is used for the heater sheaths and for other parts such as the heat duct, which operates at high temperature.

There are no radiolytic or pyrolytic composition products from these materials.

6.2.5.8 References

1. NRC Regulations:

Regulatory Guide 1.7, Rev. 2, November 1978.

Standard Review Plan 6.2.5, "Combustible Gas Control in Containment," July 1981.

6.2-61

- Branch Technical Position CSB 6-2, "Control of Combustible Gas Concentration in Containment Following a Loss-of-Coolant Accident."
- 2. Fletcher, W. D., Bell, M. J., and Picone, L. F., "Post-LOCA Hydrogen Generation in PWR Containments," <u>Nuclear Technology 10</u>, pp 420-427, 1971.
- 3. Zittel, H. E., and Row, T. H., "Radiation and Thermal Stability of Spray Solutions," Nuclear Technology 10, pp 436-443, 1971.
- 4. Allen, A. O., <u>The Radiation Chemistry of Water and Aqueous Solutions</u>, Princeton, N.J., Van Nostrand, 1961.
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- 6. Deleted.
- 7. Cottrell, W. B., "ORNL Nuclear Safety Research and Development Program Bi-Monthly Report for July-August 1968," <u>ORNL-TM-2368</u>, November 1968.
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- 10. Alan R. Barton Nuclear Plant, Preliminary Safety Analysis Report, paragraph 6.2.5.3.3.
- 11. "Natural Transport Effects on Fission Product Behavior in the Containment Systems Experiment," BNWL-1457, December 1970.
- 12. "Nuclear Safety Quarterly Report-July, August, September, and October 1967," <u>BNWL-</u>754, June 1968.
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- 14. "Hydrogen Mixing Within the Drywell Prior to Drywell Containment Mixing System Actuation," <u>GESSAR</u>, Section 6.2.5.3.3.1, March 1975.
- 15. Roberts, A., <u>et al.</u>, "Methane Layering in Mine Airways," <u>Colliery Guardian</u>, October 1962.
- 16. United States Atomic Energy Commission, 169th General Meeting of the Advisory Committee on Reactor Safeguards, May 9, 1974.
- 17. Wilson, J. F., "Electric Hydrogen Recombiner for Water Reactor Containments," <u>WCAP-7709-L</u> (Proprietary), July 1971, and <u>WCAP-7820</u> (Nonproprietary), December 1971.
- 18. Wilson, J. F., "Electric Hydrogen Recombiner for PWR Containments-Final Development Report," <u>WCAP-7709-L</u>, Supplement 1 (Proprietary), and <u>WCAP-7820</u>, Supplement 1 (Nonproprietary), April 1972.
- 19. Wilson, J. F., "Electric Hydrogen Recombiner for PWR Containments-Equipment Qualification Report," <u>WCAP-7709-L</u>, Supplement 2 (Proprietary), and <u>WCAP-7820</u>, Supplement 2 (Nonproprietary), September 1973.
- 20. Wilson, J. F., "Electric Hydrogen Recombiner for PWR Containments Long Term Tests," <u>WCAP-7709-L</u>, Supplement 3 (Proprietary), and <u>WCAP-7820</u>, Supplement 3 (Nonproprietary), January 1974.
- 21. Wilson, J. F., "Electric Hydrogen Recombiner for PWR Containments," <u>WCAP-7709-L</u>, Supplement 4 (Proprietary), and <u>WCAP-7820</u>, Supplement 4 (Nonproprietary), April 1974.

- 22. Wilson, J. F., "Electric Hydrogen Recombiner Special Tests," <u>WCAP-7709-L</u>, Supplement 5 (Proprietary), and <u>WCAP-7820</u>, Supplement 5 (Nonproprietary), December 1975.
- 23. Wilson, J. F., "Electric Hydrogen Recombiner IEEE 323-1974 Qualification," <u>WCAP-7709-L</u>, Supplement 6 (Proprietary), and <u>WCAP-7820</u>, Supplement 6 (Nonproprietary), October 1976.
- 24. Wilson, J. F., "Electric Hydrogen Recombiner LWR Containments Supplemental Test Number 2," <u>WCAP-7709-L</u>, Supplement 7 (Proprietary), and <u>WCAP-7820</u>, Supplement 7 (Nonproprietary), August 1977.

6.2.6 CONTAINMENT LEAKAGE TESTING^a

The reactor containment, containment penetrations, and containment isolation barriers are designed to permit periodic leakage rate testing as required by 10 CFR 50, Appendix A, General Design Criteria (GDC) 52, 53, and 54. The containment leak test requirements are outlined and the acceptance criteria for such tests are established in 10 CFR 50, Appendix J, Option B. The objective of the leakage rate testing is to ensure that the leakage from the containment is within the limits set by the Technical Specifications.

Compliance with 10 CFR 50, Appendix J, Option B, Types A, B, and C, testing is discussed in paragraphs 6.2.6.1, 6.2.6.2, 6.2.6.3, and 6.2.6.4.

6.2.6.1 Containment Integrated Leakage Rate Test (Type A Test)

The design leakage rate (L_d) for the containment is 0.2% free volume per day for the first 24 h. The actual leakage rate will be determined by using the methods and requirements of 10 CFR 50, Appendix J, Option B, for Type A tests.

The acceptance criteria specified in Appendix J for the integrated leakage rate test (ILRT) includes a margin for possible deterioration of the containment leakage integrity during the service intervals between tests. The measured leakage rate (L_{am}) will be less than 0.75 of the maximum allowable leakage rate value L_a .

6.2.6.1.1 ILRT Pretest Requirements

Several pretest requirements are to be met before the ILRT is performed. A general inspection of the accessible nonconcrete interior and exterior surfaces of the containment structures and components for any evidence of structural deterioration which may affect either the containment structural integrity or leaktightness will be made. The visual examination of containment concrete surfaces intended to fulfill the requirements of 10 CFR 50, Appendix J, Option B testing, will be performed in accordance with the requirements of and frequency specified by ASME Section XI Code, Subsection IWL, except where relief has been authorized by the NRC. At the discretion of the licensee, the containment concrete visual examinations may be performed during either power operation, e.g., performed concurrently with other containment inspection-related activities such as tendon testing, or during a maintenance/refueling outage.

^a The 10 CFR 50 Appendix J Program is credited as a license renewal aging management program (see subsection 19.2.29).

Any evidence of structural deterioration will be evaluated and corrected if necessary before the Type A test is performed.

Systems that are required to maintain the plant in a safe condition during the test, such as the nuclear service cooling water lines to the containment air coolers, are operable in their normal mode and need not be vented.

Systems that are normally filled with water and operating under post-accident conditions, such as the containment heat removal system, need not be vented or drained. Systems which are not vented to containment or drained during Type A tests are identified in table 6.2.6-1.

The steam generator tubes and shell and the associated piping systems passing through the containment liner are considered an extension of the containment. Therefore, the secondary side of the steam generator and connecting systems are not vented to the containment atmosphere. However, since the secondary side of the steam generator is a valid leak path, the main steam lines are vented outside of containment. The penetrations associated with the secondary side of the steam generator are identified in figure 6.2.4-1.

The reactor coolant drain tank, pressurizer relief tank, and accumulator tanks are vented to the containment atmosphere. This is done to protect the tanks from the external pressure of the test and to preclude leakage to or from the tanks which could affect the accuracy of the test results.

An equipment protection list is provided in the ILRT procedure to identify any components that cannot withstand the ILRT test pressure.

During preoperational testing, a structural integrity test (SIT) is performed. The SIT is a pressure test conducted to verify that the containment structural response due to the induced load is consistent with the predicted behavior. Paragraph 3.8.1.7 describes the SIT deflection measurements and concrete crack inspections.

Following the SIT, the preoperational ILRT is performed.

6.2.6.1.2 ILRT Test Method

The ILRT will be conducted in accordance with 10 CFR 50 Appendix J, Option B. A short duration ILRT may be performed in accordance with Bechtel's Topical Report BN-TOP-1. The test procedure used during the preoperational Type A test is described in chapter 14. Drawing 1X4DB132 shows the test arrangement for a Type A test. Two permanent flowthrough tubes have been added in a spare electrical penetration (electrical penetration number 31) to be utilized for the flow verification and pressure sensing lines. After completion of the ILRT, the flowthrough tubes are sealed and the spare electrical penetration is restored to its original configuration and Type B tested. For penetrations which are exempt from Type B or C tests the leakage testing is accomplished by the Type A test, as noted in table 6.2.4-1.

Containment dry bulb temperature, pressure, and dewpoint temperature are periodically monitored during the test. These data are analyzed as they are taken so that the leakage rate and its statistical significance are known as the test progresses. Once the leakage rate has been found with sufficient accuracy, a known additional leak is imposed and the measurements are continued, giving additional verification of the leakage rate.

6.2.6.2 Containment Penetration Leakage Rate Tests (Type B Tests)

Containment penetrations whose design incorporates resilient seals, gaskets, or sealant compounds; airlocks and lock-door seals; equipment and access hatch seals; and electrical canister and modular type penetrations receive a preoperational and periodic Type B leakage rate test in accordance with 10 CFR 50, Appendix J, Option B.

Electrical penetrations are of modular- or canister-type design (66 modular and 6 canister) and their leakage testing provisions were designed and initially tested to meet the requirements of Institute of Electrical and Electronics Engineers (IEEE) 317. Each of the 66 modular-type electrical penetrations consists of a single header plate sealed to a nozzle on the containment exterior by double O-rings with interspace connection. Feedthrough modules carrying conductors are sealed into the header plate by a metal compression fitting assembly with interspace connection. The feedthrough conductors are sealed to the feedthrough module by double, high-temperature, thermoplastic seals with interspace connection. The seal interspaces (header plate O-rings, feedthrough module compression fittings, and thermoplastic seals) are pressurized with nitrogen; the pressure is periodically monitored to detect leakage. Each of the canister-type electrical penetrations is similar in design to the modular type, except that the canister type extends through the length of the nozzle with header plates at each end, through which the conductor modules are sealed and terminate in electrical ceramic bushings. Each of the 72 electrical penetrations is provided with a local pressure gauge.

Expansion bellows utilized on the fuel transfer tube penetration accommodate relative movement between the refueling pool liner and the containment penetration and do not form part of the containment pressure boundary. Expansion bellows utilized on the containment emergency sump suction line guard pipes accommodate relative movement between the auxiliary and fuel handling buildings with respect to the containment and do not form part of the containment pressure boundary.

The equipment hatch, personnel lock, and escape hatch doors are fitted with double seals with an interspace test connection.

The test connection for the equipment hatch is located inside the containment. Test connections for the personnel lock and escape hatch doors are located such that testing is accomplished without entering the containment. Two pressure gauges are provided, one inside the lock which penetrates the bulkhead at the inner airlock door to measure containment pressure and one located outside the airlock to penetrate the bulkhead at the outer airlock door to read lock pressure. The handwheel shafts are provided with double seals and test connections where the shafts penetrate the airlock bulkheads.

The containment leak rate penetrations (table 6.2.4-1, penetrations 64A, 64B, 68, and 87) are provided with blind flanges that have testable double O-ring seals and are Type B tested. The eddy current/sludge lancing penetrations (table 6.2.4-1, penetrations 5, 55, and 90) are provided with blind flanges that have O-ring type joints per ANSI B16.5 with valve test connections and are Type B tested.

Type B tests are conducted at calculated peak post-LOCA containment internal pressure (P_a). The acceptance criteria and leakage rate limits are given in the Technical Specifications. Test methods and equipment are described in paragraph 6.2.6.3.

6.2.6.3 <u>Containment Isolation Valve Leakage Rate Tests</u>

Containment isolation valves are Type C tested in accordance with 10 CFR 50, Appendix J, Option B.

The process piping, instrumentation tubing, and personnel access penetrations are listed in table 6.2.4-1. Figure 6.2.4-1 shows the location of all test vent and drain connections and the normal direction of flow.

The CIVs for each piping penetration and process isolation valve for those piping penetrations where no GDC is applicable are tabulated in table 6.2.4-1, together with the test type.

Type B and C tests are performed by local pressurization utilizing either the pressure decay or flowmeter method. For the pressure decay method, the test volume is pressurized with air or nitrogen to at least P_a. The rate of decay of pressure of the known free air test volume is monitored to calculate the leakage rate. In the flowmeter method, pressure is maintained in the test volume by makeup air, nitrogen, or water (if applicable) through a calibrated flowmeter. The flowmeter fluid flowrate is the isolation valve leakage rate.

The leakage will be measured in the same direction as would occur in an accident, unless it can be determined that leakage measured in a different direction will provide an equivalent or more conservative result. The test medium used for pressurization is determined by the valve's post-accident condition. Valves which could be exposed to the containment atmosphere subsequent to an accident will be tested with air or nitrogen. Valves which are in lines designated to be filled with a liquid for at least 30 days subsequent to an accident may be leakage-rate tested with water.

Type C testing of the safety injection lines, residual heat removal lines, high head safety injection lines of the chemical and volume control system, RCP seal injection lines, the containment emergency sump lines to the residual heat removal and containment spray pumps, and nuclear service cooling water lines to and from the containment fan coolers is not performed. The justification for this is that these valves are either normally open at the time of a LOCA or are opened at some time after the accident to effect immediate and long term core cooling. These systems are closed systems outside containment except for the NSCW system which is a closed system inside containment, designed and constructed to ASME III. Class 2 and Seismic Category I requirements, and as such they do not constitute a potential containment atmosphere leak path during or following a loss-of-coolant accident with a single active failure of a system component. Should the valves leak slightly when closed, the fluid seal within the pipe or the closed piping system outside/inside containment would preclude release of containment atmosphere to the environs. Furthermore, inservice testing and inspection of these isolation valves and the associated piping system outside the containment is performed periodically under the inservice inspection requirements of ASME XI as described in subsection 3.9.6 and section 6.6. During normal operation, the systems are water filled, and degradation of valves or piping is readily detected. Containment penetrations not vented to containment or drained during Type A testing are identified in table 6.2.6-1.

The steam generator and associated secondary system piping that form the primary barrier to the outside, much the same as the containment liner plate, are subjected to Type A test as shown in table 6.2.4-1. The barriers against fission product release to the environment are the steam generator tubes and piping associated with the steam generators.

Containment pressure monitoring lines are considered an extension of the containment boundary, and therefore the isolation valves are not Type C tested.

Isolation valves will be positioned to their post accident position by the normal method with no accompanying adjustments. Exercising valves for the purpose of improving leakage performance shall not be permitted.

For larger test volumes, a pressure decay method may be utilized to determine the leakage rate. The makeup flow rate or the pressure decay methods or other proven techniques may be used to determine the leakage.

The total leakage rate for Type B and C tests must be less than 0.6 La.

The criteria for determining the direction in which the test pressure is applied to the isolation valves are as follows:

A. Gate Valves

- 1. Parallel Disc
- a. Test in the design basis accident (DBA) direction.
- b. Testing can be performed between the discs if a test connection or drain is provided in the valve design.
- 2. Flexible Wedge
 - a. Test in the DBA direction.
 - b. Testing can be performed between the wedge sections if a test connection or drain is provided in the valve design.
- 3. Solid Wedge
 - a. Test in the DBA direction.
- B. Globe Valves

If the DBA flow direction is over the disc (flow to close), the valve may be tested in the reverse direction. However, if the DBA flow direction is under the disc (flow to open), then the valve must be tested in this direction.

C. Butterfly Valves

Test in the DBA direction for Type C tests.

D. Flanges

Test in either direction.

The leakage rate test acceptance criteria for penetrations and isolation valves subject to Type B and C tests are given in the Technical Specifications.

6.2.6.4 Scheduling and Reporting of Periodic Tests

Type A, B, and C tests are conducted at the intervals specified in the containment leakage testing program as specified in the Technical Specifications. These intervals are in accordance with 10 CFR 50, Appendix J, Option B, with approved exceptions.

A post outage report will be prepared, presenting the results of the previous cycle's Type B and Type C tests; and Type A, Type B, and Type C tests if performed during that outage. Sufficient documentation will be collected and retained so that the effectiveness of the implementation of the containment leakage testing program can be reviewed and determined. The technical contents of the report will be available onsite for NRC review.

The preoperational test report contains a schematic of the leakage measuring system, instrumentation used, supplemental test method, test program, and analysis and interpretation of the leakage test data for the Type A test.

6.2.6.5 <u>Special Testing Requirements</u>

VEGP does not have a subatmospheric containment or a secondary containment, hence there are no special testing requirements beyond those delineated in paragraphs 6.2.6.1 through 6.2.6.4.

6.2.7 FRACTURE PREVENTION OF CONTAINMENT PRESSURE BOUNDARY

In accordance with General Design Criterion (GDC) 51, the reactor containment pressure boundary is designed with sufficient margin to ensure that under operating, maintenance, testing, and design basis accident conditions, its ferritic materials behave in a nonbrittle manner and the probability of rapidly propagating fracture is minimized.

For VEGP, the reactor containment pressure boundary components with ferritic materials are:

- Containment liner plate.
- Containment penetration sleeve assemblies.
- Equipment hatch.
- Personnel lock.
- Escape lock.
- Flued heads.
- Containment isolation boundary piping.
- Containment isolation boundary valves.

6.2.7.1 Design Bases

- A. In accordance with GDC 1, the containment pressure boundary is designed to quality standards commensurate with the safety function performed.
- B. In accordance with GDC 16, the containment pressure boundary is designed to provide a barrier against the uncontrolled release of radioactivity to the environment.

6.2.7.2 Specifications for Ferritic Materials

A. Containment Liner Plate

Since the containment liner plate is only 1/4 in. thick, it is exempt from fracture toughness testing. However, liner plate greater than 1/4 in. thick (referred to as thickened liner plate) satisfies the fracture toughness test requirements presented in NE-2320 of Subsection NE of the following code editions and addenda:

- 1. All thickened liner plate up to the basemat liner plate at el 169 ft 0 in., 1971 edition through summer 1973 addenda.
- 2. All thickened liner plate above el 169 ft 0 in., 1974 edition through summer 1975 addenda.

B. Penetration Sleeve Assemblies

The penetration sleeve assemblies are composed of the sleeve, thickened liner plate (as reinforcing), anchor rings and stiffeners, gusset plates, and cap plates for spare penetrations. The penetration sleeve assemblies for the electrical penetrations contain a weld neck flange with captive nuts attached to receive the electrical header plates. Sleeves are fabricated from SA-516 Grade 70 plates and weld neck flanges are steel forgings of SA-105 material. The impact requirements for fracture toughness for penetration sleeve assemblies in the containment shell are in accordance with American Society of Mechanical Engineers (ASME) Section III, Division 1, Subsection NE, 1974 edition through summer 1975 addenda. Penetration sleeves in the containment basemat are in accordance with ASME Section III, Division 1, Subsection NE, 1971 edition through summer 1973 addenda.

C. Equipment Hatch, Personnel Lock, and Escape Lock

These items are fracture toughness tested in accordance with ASME Section III, Division 1, Subsection NE, 1974 edition through summer 1975 addenda.

D. Flued Heads

The flued heads are fracture toughness tested in accordance with ASME Section III, Division 1, Subsection NC, 1977 through summer 1978 addenda.

E. Containment Pressure Boundary Piping

Those portions of the containment pressure boundary piping comprising the main steam and feedwater system are designed and fracture toughness tested in accordance with ASME Section III, Division 1, Subsection NC, 1974 edition through summer 1975 addenda. The remaining containment pressure boundary piping is designed in accordance with ASME III, 1974 edition through summer 1975 addenda, without fracture toughness testing. The fracture toughness of these items is further addressed in paragraph 6.2.7.4.

The configuration of containment pressure boundary piping is shown in figure 6.2.4-1.

F. Containment Pressure Boundary Valves

Pressure boundary materials of the main steam isolation valves and main feedwater isolation valves are fracture toughness tested in accordance with Subarticle NC-2300 of ASME Section III, Division 1, Subsection NC, 1977 edition. For other containment pressure boundary valves in the balance of plant scope, the specific code edition and addenda to which the valves were procured did not require fracture toughness testing. These valves were procured in accordance with the 1974 edition up to and including the summer 1975 addenda of the ASME Code. The fracture toughness of these items is further addressed in paragraph 6.2.7.4.

Containment isolation valves within the nuclear steam supply system scope are fabricated of austenitic stainless steel or carbon steel and therefore need not be fracture toughness tested.

6.2.7.3 Documentation

A. ASME Code Data Reports

ASME Code Data Reports are prepared by the cognizant organization which applies the appropriate "N" type symbol to certify that the design, fabrication, installation, inspection, testing, and stamping are in accordance with ASME III for containment pressure boundary systems.

B. Certified Material Test Reports

Certified material test reports, when applicable, are provided to document the actual results of required chemical analyses, tests, examinations, and weld repairs of materials comprising the containment pressure boundary.

C. Other Documents

Drawings and related supplemental information are available as applicable to convey information necessary to fabricate, install, test, etc., the containment pressure boundary piping system.

6.2.7.4 Evaluation

The containment isolation boundary piping and balance of plant containment isolation boundary valves were procured to meet the minimum requirements of the 1974 edition through summer 1975 addenda of the ASME Code, which did not require impact testing. The specific requirements of NX-2300 of this code effective date are that the Design Specification shall state whether or not impact testing is required. Due to the exclusion criteria of subparagraph NX-2300 of the code, the piping and valves discussed in Sections 6.2.7.2.E and F did not require impact testing.

The containment pressure boundary piping and valves meet or exceed the minimum ASME Code requirements for design, materials, fabrication, examination, testing, and code stamping.

TABLE 6.2.1-1

CONTAINMENT DESIGN LIMITS AND CALCULATED CONTAINMENT PEAK PRESSURE AND TEMPERATURE

<u>Break</u>	Peak Pressure (psig)	Available Margin (psi)	Peak Temperature <u>(°F)</u>	
Primary Side Ruptures				
Double-ended Pump Suction, Minimum Safety Injection	35.8	16.2	247	
Double-ended Pump Suction, Maximum Safety Injection	34.7	17.3	247	1
Double-ended Hot Leg	35.6	16.4	248	

Containment Design Pressure

52 psig

-3 psig

<u>Containment Atmosphere Design</u> <u>Temperature</u>

381°F^(a)

a. A peak containment atmosphere temperature of 381°F was used in calculating the thermal gradients across the containment wall.

TABLE 6.2.1-2 ASSUMPTIONS FOR CONTAINMENT ANALYSIS - PART 1

Service water temperature (°F)		95
Refueling water temperature (°F)		130
Refueling water storage tank volume (gal) (deliverable volume)		580, 497
Initial Containment		
Temperature (°F)		120
Initial pressure (psia)		17.7
Initial relative humidity (%)		20
	Net free volume (ft ³)	2.75 x 10 ⁶

TABLE 6.2.1-3 ASSUMPTIONS FOR CONTAINMENT ANALYSIS - PART 2

		Accident Analysis	<u>Design</u>
Fa	an coolers		
	Number operating	4	8
	Flowrate per fan cooler (ft³/min)	43,500 (slow speed)	97,000 (full speed)
	Heat removal capacity per fan cooler (Btu/h)	See table 6.2.2-2 (NSCW Temperature 95°F)	2.605 x 10 ⁶
S	oray pumps		
	Number available	1	2
	Flowrate per pump (gal/min)	2597	3200

TABLE 6.2.1-4 (SHEET 1 of 2)

CONTAINMENT STRUCTURAL HEAT SINKS

Passive Heat Sinks

Heat Transfer Area Ti								
Wal	I Description	<u>(ft²)</u>	<u>Material</u>	<u>(ft)</u>				
1.	Dome	29260	Epoxy Inorganic zinc paint	0.00025 0.0002083				
			Carbon steel Concrete	0.02083 3.30				
2.	Shell	65181	Inorganic zinc paint	0.0002083				
			Carbon steel Concrete	0.02083 3.75				
3.	Miscellaneous interior concrete	15971	Epoxy Concrete	0.00154 2.82				
4.	Primary shields	2106	Inorganic zinc paint	0.0002083				
			Carbon steel Epoxy Concrete	0.05795 0.001125 1.5				
5.	Mechanical equipment	209	Stainless steel	0.0158				
6	Refueling canal wall	6455	Stainless steel Epoxy Concrete	0.02083 0.001125 4.133				
7.	Miscellaneous interior concrete	4693	Epoxy Concrete	0.000042 2.893				
8.	Miscellaneous interior concrete	16147	Epoxy Concrete	0.01125 2.371				
9.	Miscellaneous interior concrete	58883	Epoxy Concrete	0.001125 1.4705				

TABLE 6.2.1-4 (SHEET 2 of 2)

		Heat Transfer Area		Thickness
Wall	<u>Description</u>	<u>(ft²)</u>	<u>Material</u>	<u>(ft)</u>
10.	Refueling canal slab	1255	Stainless steel Concrete	0.02083 4.0
11.	Structural steel	192088	Galvanizing Carbon steel	0.00033 0.0141
12.	Structural steel	183286	Inorganic zinc	0.000208
			paint Carbon steel	0.0263
13.	Piping	72630	Inorganic zinc	0.000208
			paint Carbon steel	0.03676
14.	Piping	34158	Stainless steel	0.0163
15.	Piping	1395	Stainless steel	0.0340
16.	Mechanical equipment	98158	Inorganic zinc	0.000208
			paint Carbon steel	0.142
17.	Mechanical equipment	24750	Inorganic zinc	0.000208
			paint Carbon steel	0.0208
18.	Miscellaneous steel	40599	Inorganic zinc	0.000208
			paint Carbon steel	0.01288
19.	Basement	13702	Inorganic zinc	0.000208
			paint Carbon steel Concrete	0.02083 10.5
20.	Reactor cavity	4263	Inorganic zinc	0.000208
			paint Carbon steel Concrete	0.02083 8.0

TABLE 6.2.1-5
THERMOPHYSICAL PROPERTIES OF CONTAINMENT HEAT SINKS

<u>Material</u>	Thermal Conductivity (Btu/h-ft-°F)	Volumetric Heat Capacity (Btu/ft³-°F)
Paint	0.9	20.0
Carbon steel	28.0	52.5
Stainless steel	9.4	60.1
Concrete	0.65	28.8
Galvanizing	64.8	40.9
Ероху	0.97	20.0

TABLE 6.2.1-6 ACCIDENT SEQUENCE FOR DOUBLE-ENDED PUMP SUCTION BREAK MINIMUM SAFETY INJECTION

	<u>Event</u>	Time of Occurrence (s)
1.	Accumulators begin injecting	14.5
2.	Containment peak pressure	17.6
3.	End of blowdown	22.0
4.	Safety injection begins	43.3
5.	Accumulator injection stopped	54.3
6.	Fan coolers start	100.8
7.	Spray start	108.0
8.	End of reflood	225.8
9.	Recirculation, injection	3952.0
10.	Recirculation, spray	5000.0

TABLE 6.2.1-7

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TABLE 6.2.1-8
POSTULATED SUBCOMPARTMENT PIPE BREAKS

Subcompartment	High-Energy Line	Break Area (in.²)	Maximum ΔP (psid)
Reactor cavity	1201-009-27.5 in.	144	192.90
Steam generator (pipe break below el 220 ft)	1201-008-31 in.	436	23.50
Steam generator (pipe break above el 220 ft)	1305-064-16 in.	144.5	6.77
Pressurizer	1201-053-14 in.	308	20.7

TABLE 6.2.1-9 (SHEET 1 OF 6)

RE/

Net Free Volume (ft³) 102.95	102.286	101.624	102.286	102.95	102.286	101.624	102.286	71.245	70.57
Calc Peak Pressure Differential (psig) 192.89	83.17	55.42	50.29	50.46	192.63	100.85	63.71	193.0	83.04
Humidity (%)	25	25	25	25	25	25	25	25	25
Initial Conditions Pressure (psi) 13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
Temperature (F)	120	120	120	120	120	120	120	120	120
Cross- Sectional Area (ft²) 28.4	28.22	28.034	28.22	28.4	28.22	28.034	28.22	34.20	33.88
Height (ft) 3.625	3.625	3.625	3.625	3.625	3.625	3.625	3.625	2.083	2.083
Description Approximately 1/8 of 3		Approximately 1/8 of the upper inspection annulus.	Approximately 1/8 of the lower inspection annulus.	Approximately 1/8 of 2 the lower inspection annulus.					
Volume No. 1	Ν	м	4	ß	ဖ	۲	ω	o	10

TABLE 6.2.1-9 (SHEET 2 OF 6)

Net Free	Volume (ft³) 69.91	70.57	71.245	70.57	69.91	70.57	9.169	9.447	9.7265	9.447	9.169	9.447
Calc Peak Pressure	Differential (psiq) 55.64	50.56	50.6	192.45	100.91	64.26	159.86	135.98	127.72	124.98	124.99	159.75
	Humidity (%) 25	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psi) 13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
	Temperature (F) 120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²) 33.56	33.88	34.20	33.88	33.56	33.88	1.287	1.33	1.365	1.326	1.287	1.33
	Height (f) (ft) (2.083 3	2.083 3	2.083 3	2.083 3	2.083 3	2.083 3	7.125	7.125	7.125	7.125	7.125	7.125
				L				×.	×.	÷		÷
	Description Approximately 1/8 of the lower inspection annulus.	Approximately 1/8 of the lower inspection annulus.	Approximately 1/8 of the upper inner cavity.	Approximately 1/8 of the upper inner cavity,	Approximately 1/8 of the upper inner cavity.	Approximately 1/8 of the upper inner cavity,	Approximately 1/8 of the upper inner cavity.	Approximately 1/8 of the upper inner cavity.				
	Volume No.	5	5	4	15	91	17	8	6	20	21	22

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TABLE 6.2.1-9 (SHEET 3 OF 6)

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TABLE 6.2.1-9 (SHEET 4 OF 6)

			Cross- Sectional	ı	Initial Conditions		Calc Peak Pressure	Net Free
Volume No. 35	<u>Description</u> Approximately 1/8 of	Height (ft) 6.45	Area (ft²) 3.101	Temperature (F) 120	Pressure (psi) 13.2	Humidity (%)	Differential (psiq) 54.65	Volume (ft³) 20.005
	the upper cavity annulus.							
36	Approximately 1/8 of the upper cavity annulus.	6.45	3.101	120	13.2	25	51.09	20.005
37	Approximately 1/8 of the upper cavity annulus.	6.45	3.101	120	13.2	25	50.69	20.005
38	Approximately 1/8 of the upper cavity annulus.	6.45	3.101	120	13.2	25	117.29	20.005
39	Approximately 1/8 of the upper cavity annulus.	6.45	3.101	120	13.2	25	98.57	20.005
40	Approximately 1/8 of the upper cavity annulus.	6.45	3.101	120	13.2	25	64.37	20.005
14	Approximately 1/8 of the lower cavity annulus.	6.45	3.101	120	13.2	25	76.9	20.005
42	Approximately 1/8 of the lower cavity annulus. annulus.	6.45	3.101	120	13.2	25	74.16	20.005
43	Approximately 1/8 of the lower cavity annulus.	6.45	3.101	120	13.2	25	48.21	20.005
4	Approximately 1/8 of the lower cavity annulus.	6.45	3.101	120	13.2	25	46.97	20.005
45	Approximately 1/8 of the lower cavity annulus.	6.45	3.101	120	13.2	25	47.11	20.005
46	Approximately 1/8 of the lower cavity annulus.	6.45	3.101	120	13.2	25	76.94	20.005

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TABLE 6.2.1-9 (SHEET 5 OF 6)

Net Free	Volume (ft³)	20.005	20.005	1032.93	1345.85	1431.16	2355.23	705.96	1008.52	821.894	530.23	917.62	755.48
Calc Peak Pressure	Differential (psiq)	74.93	51.3	12.4	11.91	11.25	10.59	10.2	9.71	69.6	9.68	8.43	6.74
6	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psi)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
	Temperature (F)	120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ff²)	3.101	3.101	121.81	274.66	301.30	303.9	70.60	100.85	52.96	22.58	83.42	94.44
	Height (ft)	6.45	6.45	8.48	4.90	4.75	7.75	10.0	10.0	15.52	23.48	11.0	8.0
	Description	Approximately 1/8 of the lower cavity annulus.	Approximately 1/8 of the lower cavity annulus.	Lower RX Cavity region el 160 ft 10 3/4 in. to 169 ft 4 1/2 in.	Lower RX cavity region el 156 ft to 160 ft 10 3/4 in.	Lower RX cavity; near guide tube; supports region 156 ft.	Located between first and second guide tube supports in lower RX cavity.	Incore instrumentation tunnel.	Instrumentation tunnel horizontal passageway.	Instrumentation tunnel access shaft	Instrumentation tunnel access shaft	Instrumentation tunnel	Instrumentation tunnel
	Volume No.	47	48	6	20	51	52	53	45	55	56	22	58

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TABLE 6.2.1-9 (SHEET 6 OF 6)

Net Free Volume (ft³)	755.48	938.39	1011.51	704.69	79,242.0	79,242.0	2.75E+06
Calc Peak Pressure Differential (psiq)	5.88	5.01	4.34	3.68	2.25	2.147	0.97
s Humidity (%)	25	25	25	25	25	25	25
Initial Conditions Femperature Pressure (F) (psi)	13.2	13.2	13.2	13.2	13.2	13.2	13.2
Temperature (F)	120	120	120	120	120	120	120
Cross- Sectional Area (ft²)	94.44	117.30	112.39	88.09	Ϋ́	₹ Z	ΑN
Height (ff)		8.0	0.6	8.0	∀ Z	∀ Z	NA
<u>Description</u>	Instrumentation tunnel.	Instrumentation tunnel.	Instrumentation tunnel.	Instrumentation tunnel.	South steam generator (SG) compartment.	North SG compartment.	Free containment.
Volume No.	29	09	61	62	63	64	65

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TABLE 6.2.1-10 (SHEET 1 OF 3)
REACTOR CAVITY FLOW MODEL: FLOW CHARACTERISTICS

(#_) 0.7725 0.7725 0.7727 0.7728 0.7728 0.7728 0.7728 0.7728 0.7728 0.7728 0.7728 0.7728 0.7728 0.7728 0.7728 0.7728 0.7728 0.7738
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TABLE 6.2.1-10 (SHEET 2 OF 3)

Contraction (K) Expansion (K) Turning and Obstruction (K) Description of Flow 7-7-17-27 14-15 √ent ⊃ath

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TABLE 6.2.1-10 (SHEET 3 OF 3)

L/A (ff ⁻¹)	00004 0071
Total $(K_{\overline{1}})$	22222222222222222222222222222222222222
Contraction (K)	00000000000000000000000000000000000000
Expansion (K)	
Turning and Obstruction (K)	00000000000000000000000000000000000000
Friction (K)	00000000000000000000000000000000000000
Flow Area (ft ⁾	221111226 12256 12
ion M <u>Unchoked</u>	****** *** * * * * * * **********
Description of Flow <u>Choked</u> Ur	××× ××× × ×××
Node Number (From-To)	88888888888888888888888888888888888888
Vent Path	\$51555455555555555555555555555555555555

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STEAM GENERATOR COMPARTMENT MODEL (PIPE BREAK BELOW EL. 220 ft): NODE CHARACTERISTICS LOOP CLOSURE WELD (336-in.² BREAK) TABLE 6.2.1-11 (SHEET 1 OF 7)

Net Free	Volume (ft³)	1316.42	2283.14	1661.50	809.05	895.04	3124.98	3721.19	3078.62	890.21	837.79
Calc Peak Pressure	Differential (psiq)	7.11	13.58	15.89	13.65	13.27	13.24	14.96	12.76	12.89	10.93
	Humidity (%)	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>-</u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	222.97	176.0	132.24	83.6	95.78	165.73	289.03	163.51	95.31	86.44
	Height (ft)	6.56	15.25	15.25	11.25	11.25	23.25	15.25	23.25	11.25	11.25
	Description	Interface of two halves of SG compartment.	Between cold leg and wall; from el 171 ft 9 in. to 187 ft.	Between RCP 4, SG 4, and wall; from el 171 ft 9 in. to 187 ft.	By SG 4; from el 171 ft 9 in. to 183 ft.	Adjacent to SG 4; from el 171 ft 9 in. to 183 ft.	Around SG 4; from el 171 ft 9 in. to 195 ft.	Between hot, cold and suction legs; from el 171 ft 9 in. to 187 ft.	Around SG 1; from el 171 ft 9 in. to 195 ft.	Adjacent to SG 1; from el 171 ft 9 in. to 183 ft.	By SG 1; from el 171 ft 9 in. to 183 ft.
	Volume No.	-	7	က	4	ည	9	-	ω	o o	10

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TABLE 6.2.1-11 (SHEET 2 OF 7)

Net Free	Volume (ft³)	1737.79	3729.58	2815.04	2400.44	766.38	892.25	887.09	796.98	1251.37	814.29	1581.52	1631.41
Calc Peak Pressure	Differential (psiq)	7.62	11.11	7.60	5.85	13.52	13.28	12.88	10.96	13.56	15.9	14.93	10.26
	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>:</u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	137.8	289.64	219.7	131.71	83.6	95.78	95.31	86.44	179.22	113.1	229.18	236.11
	Height (ft)	15.25	15.25	15.25	20.25	12.0	12.0	12.0	12.0	8.0	8.0	8.0	8.0
	Description	Between RCP 1, SG 1, and wall; from el 171 ft 9 in. to 187 ft.	Between hot, cold, and suction legs 1; from el 171 ft 9 in. to 187 ft.	Between hot leg 1 and wall; from el 171 ft 9 in. to 187 ft.	Interface of two halves of SG compartment; between quadrants 1 and 2.	Over node 4; from el 183 to 195 ft.	Over node 5; from el 183 to 195 ft.	Over node 9 in quadrant 1; from el 183 to 195 ft.	Over node 10 in quadrant 1; from el 183 to 195 ft.	Over node 2; from el 187 to 195 ft.	Over node 3; from el 187 to 195 ft.	Over-node 7; from el 187 to 195 ft.	Over node 12; from el 187 to 195 ft.
	Volume No.		2	6		15	16	17	8	19	20	21	22

TABLE 6.2.1-11 (SHEET 3 OF 7)

Net Free	Volume (ft³)	795.67	1572.59	402.0	142.3	348.52	145.4	324.23	668.07	460.28	788.23	671.4	315.67
Calc Peak Pressure	Differential (psiq)	8.56	8.61	13.01	13.11	13.2	12.77	12.06	12.06	12.73	12.98	11.55	11.14
	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>iu</u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	110.51	223.84	89.33	31.62	89.14	32.94	83.81	176.1	102.29	197.63	176.84	81.91
	Height (ft)	8.0	8.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	2.0
	Description	Over node 11; from el 187 to 195 ft.	Over node 13; from el 187 to 195 ft.	Around RCP 4; from el 195 to 200 ft.	Between RCP 4 and wall; from el 195 to 200 ft.	Between SG 4 and RCP 4; from el 195 to 200 ft.	Between SG 4 and HVAC shaft; from el 195 to 200 ft.	Between SG 4 and wall; from el 195 to 200 ft.	Between SG 4 and concrete beam; from el 195 to 200 ft.	Adjacent to node 32; from el 195 to 200 ft.	Opposite node 27; from el 195 to 200 ft.	Between SG 1 and concrete beam; from el 195 to 290 ft.	Between SG 1 and wall; from el 195 to 200 ft.
	Volume No.	23	24	25	26	27	28	59	30	31	32	33	34

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TABLE 6.2.1-11 (SHEET 4 OF 7)

Net Free	Volume (ft³)	146.0	393.9	169.15	513.74	912.67	463.95	1796.2	635.37	2577.68	419.36	96'.26
Calc Peak Pressure	Differential (psiq)	10.68	8.59	8.60	8.61	9.12	10.38	11.38	11.42	11.53	11.50	11.46
	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>c</u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120	102
Cross- Sectional	Area (ft²)	33.08	99.23	37.59	114.17	225.29	103.1	100.1	35.3	144.38	31.7	111.91
	Height (ft)	5.0	5.0	5.0	5.0	5.0	5.0	20.0	20.0	20.0	15.0	7.0
	Description	Between SG 1 and HVAC shaft; from el 195 to 200 ft.	Between SG 1 and RCP 1; from el 195 to 200 ft.	Between RCP 1 and wall; from el 195 to 200 ft.	Around RCP 1; from el 195 to 200 ft.	Opposite node 36; from el 195 to 200 ft.	Adjacent to node 39; from el 195 to 200 ft.	Between RCP 4 and northern wall of quadrant; from el 200 to 220 ft.	Between RCP 4 and SW wall of quadrant; from el 200 to 220 ft.	Between HVAC shaft, RCP 4, and SG 4: from el 200 to 220 ft.	Between SG 4 and southern wall of quadrant; from el 200 to 215 ft.	Between node 44 and quadrant 1; from el 200 to 207 ft.
	Volume No.	35	36	37	38	36	40	14	42	43	44	45

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TABLE 6.2.1-11 (SHEET 5 OF 7)

Net Free	Volume	<u>π)</u> 796.22		3015.61	1258.89	2214.86	3012.0	695.7	798.54	402.25	2640.6
Calc Peak Pressure	Differential	(psiq) 11.27		11.49	11.58	11.53	10.89	11.07	10.96	10.8	8.55
	Humidity	(%)		25	25	25	25	25	25	25	25
Initial Conditions	Pressure	(psia) 13.2		13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>i</u>	Temperature	(F)		120	120	120	120	120	120	120	120
Cross- Sectional	Area	(II.) 110.59		225.12	93.84	124.07	224.85	111.55	110.91	30.43	147.88
	Height	(II) 8.0		15.0	15.0	20.0	15.0	7.0	8.0	15.0	20.0
	Coordination	<u>Description</u> Over node 45;	from el 207 to 215 ft.	Between node 48 and quadrant 1; from el 200 to 215 ft.	Between SG 4 and northern wall, adjacent to node 49; from el 200 to 215 ft.	Adjacent to node 43; between RCP 4 and northern wall; from el 200 to 220 ft.	Between node 58 and quadrant 4; from el 200 to 215 ft.	Between node 53 and quadrant 4; from el 200 to 207 ft.	Over node 51; from el 207 to 215 ft.	Between SG 1 and southern wall of quadrant 1; from el 200 to 215 ft.	Between HVAC shaft, RCP 1, and SG 1; from el 200 to 220 ft.
	Volume	No.		47	48	49	20	51	52	53	54

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TABLE 6.2.1-11 (SHEET 6 OF 7)

Net Free	Volume (ft³)	646.32	1986.14	2488.86	1347.85	918.0	596.4	404.2	1356.2	1234.0
Calc Peak Pressure	Differential (psiq)	8.33	8.36	8.66	9.40	3.75	4.41	3.28	3.88	3.87
	Humidity (%)	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
i <u>u</u>	Temperature (F)	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	35.91	110.65	139.29	100.43	50.73	52.98	52.84	79.51	79.87
	Height (ft)	20.0	20.0	20.0	15.0	23.0	0.41	0.6	23.0	23.0
	Description	Between RCP 1 and SE wall of quadrant 1; from el 200 to 220 ft.	Between RCP 1 and northern wall of quadrant 1; from el 200 to 220 ft.	Adjacent to node 54; between RCP 1 and northern wall; from el 200 to 220 ft.	Between SG 1 and northern wall, adjacent to node 57; from el 200 to 215 ft.	NW node of left half of SG 4 doghouse; from el 215 to 238 ft.	SW node of left half of SG 4 dog- house; from el 215 to 229 ft.	Over node 60; from el 229 to 238 ft.	SE node of left half of SG 4 dog- house; from el 215 to 238 ft.	NE node of left half of SG 4 dog- house; from el 215 to 238 ft.
	Volume No.	55	56	57	28	29	09	61	62	63

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TABLE 6.2.1-11 (SHEET 7 OF 7)

	ential Volume iig) (ft³)		3.76 1356.2		3.20 404.2		11.41 838.6	7.91 838.6	•		3.06 8.36E+04	2.74 2.75E+06
Ī				25 4.12			25 11	25 7	25 8	25 6	25 3	25 2
				13.2			13.2	13.2	13.2	13.2	13.2	13.2
	l emperature (F)	120	120	120	120	120	120	120	120	120	120	120
	t Area (ff²)	79.87	79.51	52.98	52.84	50.73	ı	ı	ı	ı	ı	•
:	Height (ft)	de 23.0	de 23.0	de 14.0	o.e 9.0	de 23.0	1	1	1	1	13 - ent.	•
	Description	Symmetric of node 63 in quadrant 1.	Symmetric of node 62 in quadrant 1.	Symmetric of node 60 in quadrant 1.	Symmetric of node 61 in quadrant 1.	Symmetric of node 59 in quadrant 1.	HVAC shaft 4	HVAC shaft 1.	HVAC duct 4.	HVAC duct 1.	Quadrants 2 and 3 of SG compartment.	Containment atmosphere.
;	Volume No.	64	65	99	29	89	69	20	71	72	73	74

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TABLE 6.2.1-11A STEAM GENERATOR COMPARTMENT MODEL (PIPE BREAK ABOVE EL. 220 ft): NODE CHARACTERISTICS

Net Free	Volume (ft³)	1796	409	434	657	707	785	1039	1221	2665	1461	2978	2.75x10 ⁶
Calc Peak Pressure	Differential (psiq)	8.06	5.31	5.23	6.22	6.65	6.56	6.90	6.58	6.49	6.58	09.9	3.29
	Humidity (%)	20	20	20	20	20	20	20	20	20	20	20	20
Initial Conditions	Pressure (psia)	14.7	14.7	14.7	14.7	14.7	14.7	14.7	14.7	14.7	14.7	14.7	14.7
<u>-</u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	149.7	45.4	48.2	73.0	54.4	60.4	6.62	76.3	166.56	91.3	186.13	1
	Height (ft)	12	ത	ത	ത	13	13	13	16	16	16	16	1
	Description	Quadrant 1 from el 216 ft to 238 ft	Quadrant 2 from el 229 ft to 238 ft	Quadrant 3 from el 229 ft to 238 ft	Quadrant 4 from el 229 ft to 238 ft	Quadrant 2 from el 216 ft to 229 ft	Quadrant 3 from el 216 ft to 229 ft	Quadrant 4 from el 216 ft to 229	Quadrant 2 from el 200 ft to 216 ft	Quadrant 3 from el 200 ft to 216 ft	Quadrant 4 from el 200 ft to 216 ft	Quadrant 1 from el 200 ft to 216 ft	Containment Atmosphere
	Volume No.	-	7	ო	4	ις	O	2	∞	ത	10		12

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TABLE 6.2.1-12 (SHEET 1 OF 7)

STEAM GENERATOR COMPARTMENT MODEL (PIPE BREAK BELOW EL 220 ft): NODE CHARACTERISTICS STEAM GENERATOR INLET NOZZLE (306-in.² BREAK)

Net Free	Volume (ft³)	1316.42	2283.14	1661.50	809.05	895.04	3124.98	3721.19	3078.62	890.21	837.79
Calc Peak Pressure	Differential (psiq)	3.08	7.43	7.91	8.21	8.48	0.6	0.6	8.24	8.38	6.35
	Humidity (%)	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>:</u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	222.97	176.0	132.24	83.6	95.78	165.73	189.03	163.51	95.31	86.44
	Height (ft)	6.56	15.25	15.25	11.25	11.25	23.25	15.25	23.25	11.25	11.25
	Description	Interface of two halves of SG compartment.	Between cold leg and wall; from el 171 ft 9 in. to 187 ft.	Between RCP 4, SG 4, and wall; from el 171 ft 9 in. to 187 ft.	By SG 4; from el 171 ft 9 in. to 183 ft.	Adjacent to SG 4; from el 171 ft 9 in. to 183 ft.	Around SG 4; from el 171 ft 9 in. to 195 ft.	Between hot, cold, and suction legs; from el 171 ft 9 in. to 187 ft.	Around SG 1; from el 171 ft 9 in. to 195 ft.	Adjacent to SG 1; from el 171 ft 9 in. to 183 ft.	By SG 1; from el 171 ft 9 in. to 183 ft.
	Volume No.	-	7	т	4	co	ø	~	ω	O	0

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TABLE 6.2.1-12 (SHEET 2 OF 7)

Net Free	Volume (ft³)	1737.79	3729.58	2815.04	2400.44	766.38	892.25	887.09	796.98	1251.37	814.29	1581.52	1631.41
Calc Peak Pressure	Differential (psig)	3.90	6.57	3.95	2.71	8.19	8.46	8.82	6.25	7.43	7.98	8.92	5.82
	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
	Temperature (F)	120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	137.8	289.64	219.7	131.71	83.6	95.78	95.31	86.44	179.22	113.1	229.18	236.11
	Height (ft)	15.25	15.25	15.25	20.25	12.0	12.0	12.0	12.0	8.0	8.0	8.0	0.8
	Description	Between RCP 1, SG 1, and wall; from el 171 ft 9 in. to 187 ft.	Between hot, cold, and suction legs 1; from el 171 ft 9 in. to 187 ft.	Between hot leg 1 and wall; from el 171 ft 9 in. to 187 ft.	Interface of two halves of SG compartment; between quadrants 1 and 2.	Over node 4; from el 183 to 195 ft.	Over node 5; from el 183 to 195 ft.	Over node 9 in quadrant 1; from el 183 to 195 ft.	Over node 10 in quadrant 1; from el 183 to 195 ft.	Over node 2; from el 187 to 195 ft.	Over node 3; from el 187 to 195 ft.	Over node 7; from el 187 to 195 ft.	Over node 12; from el 187 to 195 ft.
	Volume No.	-	72	1 3	4	15	16	17	81	19	20	21	22

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TABLE 6.2.1-12 (SHEET 3 OF 7)

Net Free	Volume (ft³)	795.67	1572.59	402.0	142.3	348.52	145.4	324.23	668.07	460.28	788.23	671.4	315.67
Calc Peak Pressure	Differential (psig)	4.66	4.54	7.19	7.16	7.18	7.11	6.90	6.94	7.13	7.18	6.70	6.43
	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u> </u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	110.51	223.84	89.33	31.62	89.14	32.94	83.81	176.1	102.29	197.63	176.84	81.91
	Height (ft)	8.0	8.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0
	Description	Over node 11; from el 187 to 195 ft.	Over node 13; from el 187 to 195 ft.	Around RCP 4; from el 195 to 200 ft.	Between RCP 4 and wall; from el 195 to 200 ft.	Between SG 4 and RCP 4; from el 195 to 200 ft.	Between SG 4 and HVAC shaft; from el 195 to 200 ft.	Between SG 4 and wall; from el 195 to 200 ft.	Between SC 4 and concrete beam; from el 195 to 200 ft.	Adjacent to node 32; from el 195 to 200 ft.	Opposite node 27; from el 195 to 200 ft.	Between SG 1 and concrete beam; from el 195 to 200 ft.	Between SG 1 and wall; from el 195 to 200 ft.
	Volume No.	23	24	25	26	27	28	29	30	31	32	33	34

TABLE 6.2.1-12 (SHEET 4 OF 7)

Net Free	(ff ³)	146.0	393.9	169.15	513.74	912.67	463.95	1796.2	635.37	2577.68	419.36	96'.69
Calc Peak Pressure	(psiq)	0.0	4.67	4.54	4.53	4.87	5.85	6.12	6.10	6.22	6.23	6.39
Himidity	(%)	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	(psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
ln Tamparatura	(F)	120	120	120	120	120	120	120	120	120	120	102
Cross- Sectional	(ff ²)	33.08	99.23	37.59	114.17	225.29	103.1	100.1	35.3	144.38	31.7	111.91
	(ff)	5.0	5.0	5.0	5.0	5.0	5.0	20.0	20.0	20.0	15.0	7.0
	Description	Between SG 1 and HVAC shaft; from el 195 to 200 ft.	Between SG 1 and RCP 1; from el 195 to 200 ft.	Between RCP 1 and wall; from el 195 to 200 ft.	Around RCP 1; from el 195 to 200 ft.	Opposite node 36; from el 195 to 200 ft.	Adjacent to node 39; from el 195 to 200 ft.	Between RCP 4 and northern wall of quadrant; from el 200 to 220 ft.	Between RCP 4 and SW wall of quadrant; from el 200 to 220 ft.	Between HVAC shaft, RCP 4, and SG 4. from el 200 to 220 ft.	Between SG 4 and southern wall of quadrant; from el 200 to 215 ft.	Between node 44 and quadrant 1; from el 200 to 207 ft.
omilo//	No.	35	36	37	38	39	40	1	42	43	44	45

TABLE 6.2.1-12 (SHEET 5 OF 7)

Net Free	Volume (ff³)	796.22	3015.61	1258.89	2214.86	3012.0	695.7	798.54	402.25	2640.6
Calc Peak Pressure	Differential (psiq)	6.19	6.30	6.30	6.22	6.03	6.23	6.07	5.97	4.38
	Humidity (%)	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
	Temperature (F)	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	110.59	225.12	93.84	124.07	224.85	111.55	110.91	30.43	147.88
	Height (ft)	8.0	15.0	15.0	20.0	15.0	7.0	8.0	15.0	20.0
	Description	Over node 45; from el 207 to 215 ft.	Between node 48 and quadrant 1; from el 200 to 215 ft.	Between SG 4 and northern wall, adjacent to node 49; from el 200 to 215 ft.	Adjacent to node 43; between RCP 4 and northern wall; from el 200 to 220 ft.	Between node 58 and quadrant 4; from el 200 to 215 ft.	Between node 53 and quadrant 4; from el 200 to 207 ft.	Over node 51; from el 207 to 215 ft.	Between SG 1 and southern wall of quadrant 1; from el 200 to 215 ft.	Between HVAC shaft, RCP 1, and SG 1; from el 200 to 220 ft.
	Volume No.	46	47	88	49	20	51	52	53	54

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TABLE 6.2.1-12 (SHEET 6 OF 7)

Net Free Volume (ft³)	646.32	1986.14	2488.86	1347.85	918.0	596.4	404.2	1356.2	1234.0
Calc Peak Pressure Differential (psig)	4.23	4.27	4.47	4.99	2.10	2.33	1.94	2.17	2.15
Humidity (%)	25	25	25	25	25	25	25	25	25
Initial Conditions Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
Ini Temperature (F)	120	120	120	120	120	120	120	120	120
Cross- Sectional Area (ft²)	35.91	110.65	139.29	100.43	50.73	52.98	52.84	79.51	79.87
Height (ff)	20.0	20.0	20.0	15.0	23.0	0.41	0.0	23.0	23.0
Description	Between RCP 1 and SE wall of quadrant 1; from el 200 to 220 ft.	Between RCP 1 and northern wall of quadrant 1; from el 200 to 220 ft.	Adjacent to node 54; between RCP 1 and northern wall; from el 200 to 220 ft.	Between SG 1 and northern wall, adjacent to node 57; from el 200 to 215 ft.	NW node of left half of SG 4 doghouse; from el 215 to 238 ft.	SW node of left half of SG 4 doghouse; from el 215 to 229 ft.	Over node 60; from el 229 to 238 ft.	SE node of left half of SG 4 dog- house; from el 215 to 238 ft.	NE node of left half of SG 4 dog- house; from el 215 to 238 ft.
Volume No.	55	26	57	28	59	09	19	62	63

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TABLE 6.2.1-12 (SHEET 7 OF 7)

Net Free Volume	(ff ³)	1234.0	1356.2	596.4	404.2	918.0	838.6	838.6	1231.6	1231.6	8.36E+04	2.75E+06
Calc Peak Pressure Differential	(bisd)	2.11	2.11	2.24	1.92	2.05	90.9	3.98	3.99	3.0	1.85	1.75
Humidity	(%)	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions Pressure	(psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
In Temperature	(F)	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional Area	(ff ²)	79.87	79.51	52.98	52.84	50.73	1	1	1	1		ı
Height	(#)	23.0	23.0	14.0	0.6	23.0	ı	,	,	,		
	Description	Symmetric of node 63 in quadrant 1.	Symmetric of node 62 in quadrant 1.	Symmetric of node 60 in quadrant 1.	Symmetric of node 61 in quadrant 1.	Symmetric of node 59 in quadrant 1.	HVAC shaft 4	HVAC shaft 1.	HVAC duct 4.	HVAC duct 1.	Quadrants 2 and 3 of SG compartment.	Containment atmosphere.
Volume	No.	64	65	99	29	89	69	20	71	72	73	74

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TABLE 6.2.1-13 (SHEET 1 OF 7)

STEAM GENERATOR COMPARTMENT MODEL (PIPE BREAK BELOW EL 220 ft): NODE CHARACTERISTICS STEAM GENERATOR OUTLET NOZZLE (436-in.² BREAK)

Net Free	Volume (ft³)	1316.42	2283.14	1661.50	809.05	895.04	3124.98	3721.19	3078.62	890.21	837.79
Calc Peak Pressure	Differential (psiq)	8.71	16.79	21.33	17.26	16.41	16.27	17.45	15.72	15.85	15.98
	Humidity (%)	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u> </u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ff²)	222.97	176.0	132.24	83.6	95.78	165.73	189.03	163.51	95.31	86.44
	Height (ft)	6.56	15.25	15.25	11.25	11.25	23.25	15.25	23.25	11.25	11.25
	Description	Interface of two halves of SG compartment.	Between cold leg and wall; from el 171 ft 9 in. to 187 ft.	Between RCP 4, SG 4, and wall; from el 171 ft 9 in. to 187 ft.	By SG 4; from el 171 ft 9 in. to183 ft.	Adjacent to SG 4; from el 171 ft 9 in. to 183 ft.	Around SG 4; from el 171 ft 9 in. to 195 ft.	Between hot, cold, and suction legs; from el 171 ft 9 in. to 187 ft.	Around SG 1; from el 171 ft 9 in. to 195 ft.	Adjacent to SG 1; from el 171 ft 9 in. to 183 ft.	By SG 1; from el 171 ft 9 in. to 183 ft.
	Volume No.	-	7	ო	4	വ	O	~	∞	O	0

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TABLE 6.2.1-13 (SHEET 2 OF 7)

Net Free	Volume (ft³)	1737.79	3729.58	2815.04	2400.44	766.38	892.25	887.09	796.98	1251.37	814.29	1581.52	1631.41
Calc Peak Pressure	Differential (psiq)	9.83	13.90	9.81	7.53	17.17	16.39	15.90	15.94	16.83	24.79	17.57	12.96
	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u> </u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	137.8	289.64	219.7	131.71	83.6	95.78	95.31	86.44	179.22	113.1	229.18	236.11
	Height (ft)	15.25	15.25	15.25	20.25	12.0	12.0	12.0	12.0	8.0	8.0	8.0	8.0
	Description	Between RCP 1, SG 1, and wall; from el 171 ft 9 in. to 187 ft.	Between hot, cold, and suction legs 1; from el 171 ft 9 in. to 187 ft.	Between hot leg 1 and wall; from el 171 ft 9 in. to 187 ft.	Interface of two halves of SG compartment; between quadrants 1 and 2.	Over node 4; from el 183 to 195 ft.	Over node 5; from el 183 to 195 ft.	Over node 9 in quadrant 1; from el 183 to 195 ft.	Over node 10 in quadrant 1; from el 183 to 195 ft.	Over node 2; from el 187 to 195 ft.	Over node 3; from el 187 to 195 ft.	Over-node 7; from el 187 to 195 ft.	Over node 12; from el 187 to 195 ft.
	Volume No.		5	5	4	15	16	17	81	61	20	21	22

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TABLE 6.2.1-13 (SHEET 3 OF 7)

Volume		Heiaht	Cross- Sectional Area	Initi Temperature	Initial Conditions Pressure	Humidity	Calc Peak Pressure Differential	Net Free Volume
	<u>Description</u>	(#)	(ff ²)	(F)	(psia)	(%)	(pisq)	(ff ³)
	Over node 11; from el 187 to 195 ft.	8.0	110.51	120	13.2	25	11.15	795.67
	Over node 13; from el 187 to 195 ft.	8.0	223.84	120	13.2	25	11.08	1572.59
	Around RCP 4; from el 195 to 200 ft.	5.0	89.33	120	13.2	25	16.26	402.0
	Between RCP 4 and wall; from el 195 to 200 ft.	5.0	31.62	120	13.2	25	16.54	142.3
	Between SG 4 and RCP 4; from el 195 to 200 ft.	5.0	89.14	120	13.2	25	17.76	348.52
	Between SG 4 and HVAC shaft; from el 195 to 200 ft.	5.0	32.94	120.	13.2	25	16.72	145.4
	Between SG 4 and wall; from el 195 to 200 ft.	5.0	83.81	120	13.2	25	15.26	324.23
	Between SG 4 and concrete beam; from el 195 to200 ft.	5.0	176.1	120	13.2	25	15.22	668.07
	Adjacent to node 32; from el 195 to 200 ft.	5.0	102.29	120	13.2	25	15.87	460.28
	Opposite node 27; from el 195 to200 ft.	5.0	197.63	120	13.2	25	16.20	788.23
	Between SG 1 and concrete beam; from el 195 to 200 ft.	5.0	176.84	120	13.2	25	14.66	671.4
	Between SG 1 and wall; from el195 to 200 ft.	5.0	81.91	120	13.2	25	14.71	315.67

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TABLE 6.2.1-13 (SHEET 4 OF 7)

	Height	Cross- Sectional Area	In Temperature	Initial Conditions Pressure	Humidity	Calc Peak Pressure Differential	Net Free Volume
Description	(()	(# ²)	(F)	(psia)	(%)	(bisd)	(#3)
Between SG 1 and HVAC shaft; from el 195 to 200 ft.	5.0	33.08	120	13.2	25	14.36	146.0
Between SG 1 and RCP 1; from el195 to 200 ft.	5.0	99.23	120	13.2	25	11.14	393.9
Between RCP 1 and wall; from el 195 to 200 ft.	5.0	37.59	120	13.2	25	11.18	169.15
Around RCP 1; from el 195 to 200 ft.	5.0	114.17	120	13.2	25	11.10	513.74
Opposite node 36; from el 195 to200 ft.	5.0	225.29	120	13.2	25	11.55	912.67
Adjacent to node 39; from el 195 to 200 ft.	5.0	103.1	120	13.2	25	13.20	463.95
Between RCP 4 and northern wall of quadrant; from el 200 to 220 ft.	20.0	100.1	120	13.2	25	14.58	1796.2
Between RCP 4 and SW wall of quadrant; from el 200 to 220 ft.	20.0	35.3	120	13.2	25	14.54	635.37
Between HVAC shaft, RCP 4, and SG 4. from el 200 to 220 ft.	20.0	144.38	120	13.2	25	14.71	2577.68
Between SG 4 and southern wall of quadrant; from el 200 to 215 ft.	15.0	31.7	120	13.2	25	14.87	419.36
Between node 44 and quadrant 1; from el 200 to 207 ft.	7.0	111.91	102	13.2	25	14.69	96.769

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TABLE 6.2.1-13 (SHEET 5 OF 7)

Net Free	Volume (ft³)	796.22	3015.61	1258.89	2214.86	3012.0	2.569	798.54	402.25	2640.6
Calc Peak Pressure	Differential (psig)	14.48	14.63	14.74	14.7	13.96	14.37	14.19	14.13	11.02
	Humidity (%)	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u> </u>	Temperature (F)	120	120	120	120	120 °	120	120	120	120
Cross- Sectional	Area (ft²)	110.59	225.12	93.84	124.07	224.85	111.55	110.91	30.43	147.88
	Height (ft)	8.0	15.0	15.0	20.0	15.0	7.0	8.0	15.0	20.0
	Description	Over node 45; from el 207 to 215 ft.	Between node 48 and quadrant 1; from el 200 to 215 ft.	Between SG 4 and northern wall, adjacent to node 49; from el 200 to 215 ft.	Adjacent to node 43; between RCP 4 and northern wall; from el 200 to 220 ft.	Between node 58 and quadrant 4; from el 200 to 215 ft.	Between node 53 and quadrant 4; from el 200 to 207 ft.	Over node 51; from el 207 to 215 ft.	Between SG 1 and southern wall of quadrant 1; from el 200 to 215 ft.	Between HVAC shaft, RCP 1, and SG 1,; from el 200 to 220 ft.
	Volume No.	46	47	8	64	20	51	52	53	54

TABLE 6.2.1-13 (SHEET 6 OF 7)

Net Free	Volume (ft³)	646.32	1986.14	2488.86	1347.85	918.0	596.4	404.2	1356.2	1234.0
Calc Peak Pressure	Differential (psiq)	10.75	10.77	11.12	12.04	4.45	5.28	3.85	4.62	4.60
	Humidity (%)	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>iii</u>	Temperature (F)	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	35.91	110.65	139.29	100.43	50.73	52.98	52.84	79.51	79.87
	Height (ft)	20.0	20.0	20.0	15.0	23.0	14.0	0.6	23.0	23.0
	Description	Between RCP 1 and SE wall of quadrant 1; from el 200 to 220 ft.	Between RCP 1 and northern wall of quadrant 1; from el 200 to 220 ft.	Adjacent to node 54; between RCP 1 and northern wall; from el 200 to 220 ft.	Between SG 1 and northern wall, adjacent to node 57; from el 200 to 215 ft.	NW node of left half of SG 4 doghouse; from el 215 to 238 ft.	SW node of left half of SG 4 dog house; from el 215 to 229 ft.	Over node 60; from el 229 to 238 ft.	SE node of left half of SG 4 dog house; from el 215 to 238 ft.	NE node of-left half of SG 4 dog house; from el 215 to 238 ft.
	Volume No.	22	26	22	28	26	09	61	62	63

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TABLE 6.2.1-13 (SHEET 7 OF 7)

Calc Peak Pressure Differential (psig) (ft ³)	4.47 1234.0	4.48 1356.2	4.96 596.4	3.76 404.2	4.28 918.0	14.63 838.6	10.20 838.6	11.34 1231.6	7.88 1231.6	3.58 8.36E+04	3.15 2.75E+06
Humidity (%)	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
Temperature (F)	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional Area (ft²)	79.87	79.51	52.98	52.84	50.73	1	ı	1	1	ı	
Height (ft)	23.0	23.0	14.0	0.6	23.0	•	ı	1		1	ı
Description	Symmetric of node 63 in quadrant 1.	Symmetric of node 62 in quadrant 1.	Symmetric of node 60 in quadrant 1.	Symmetric of node 61 in quadrant 1.	Symmetric of node 59 in quadrant 1.	HVAC shaft 4	HVAC shaft 1.	HVAC duct 4.	HVAC duct 1.	Quadrants 2 and 3 of SG compartment.	Containment
Volume No.	64	65	99	29	89	69	70	7.1	72	73	74

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TABLE 6.2.1-14 (SHEET 1 OF 7)

STEAM GENERATOR COMPARTMENT MODEL (PIPE BREAK BELOW EL. 220 ft): NODE CHARACTERISTICS REACTOR COOLANT PUMP OUTLET NOZZLE (236-in.² BREAK)

Net Free	Volume (ft³)	1316.42	2283.14	1661.50	809.05	895.04	3124.98	3721.19	3078.62	890.21	837.79
Calc Peak Pressure	Differential (psiq)	7.62	14.32	13.42	12.51	12.36	12.34	14.33	11.92	12.0	10.56
	Humidity (%)	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
u	Temperature (F)	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft^2)	222.97	176.0	132.24	83.6	95.78	165.73	189.03	163.51	95.31	86.44
	Height (ff)	6.56	15.25	15.25	11.25	11.25	23.25	15.25	23.25	11.25	11.25
	Description	Interface of two halves of SG compartment.	Between cold leg and wall; from el 171 ft 9 in. to 187 ft.	Between RCP 4, SG 4, and wall; from el 171 ft 9 in. to 187 ft.	By SG 4; from el 171 ft 9 in. to 183 ft.	Adjacent to SG 4; from el 171 ft 9 in. to 183 ft.	Around SG 4; from el 171 ft 9 in. to 195 ft.	Between hot, cold, and suction legs; from el 171 ft 9 in. to 187 ft.	Around SG 1; from el 171 ft 9 in. to 195 ft.	Adjacent to SG 1; from el 171 ft 9 in. to 183 ft.	By SG 1; from el 171 ft 9 in. to 183 ft.
	Volume No.	-	7	m	4	ιΩ	9	_	œ	O	0

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TABLE 6.2.1-14 (SHEET 2 OF 7)

Net Free	Volume (ft³)	1737.79	3729.58	2815.04	2400.44	766.38	892.25	887.09	796.98	1251.37	814.29	1581.52	1631.41
Calc Peak Pressure	Differential (psiq)	7.35	10.44	7.34	5.79	12.41	12.32	12.0	10.55	14.37	13.37	14.25	69.6
:	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
	Temperature (F)	120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	137.8	289.64	219.7	131.71	83.6	95.78	95.31	86.44	179.22	113.1	229.18	236.11
:	Height (ft)	15.25	15.25	15.25	20.25	12.0	12.0	12.0	12.0	8.0	8.0	8.0	8.0
	Description	Between RCP 1, SG 1, and wall; from el 171 ft 9 in. to 187 ft.	Between hot, cold, and suction legs 1; from el 171 ft 9 in. to 187 ft.	Between hot leg 1 and wall; from el 171 ft 9 in. to 187 ft.	Interface of two halves of SG compartment; between quadrants 1 and 2.	Over node 14; from el 183 to 195 ft.	Over node 5; from el 183 to 195 ft.	Over node 9 in quadrant 1; from el 183 to 195 ft.	Over node 10 in quadrant 1; from el 183 to 195 ft.	Over node 2; from el 187 to 195 ft.	Over node 3; from el 187 to 195 ft.	Over node 7; from el 187 to 1'95 ft.	Over node 12; from el 187 to 195 ft.
:	Volume No.		75	52	4	15	16	17	8	19	20	21	22

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TABLE 6.2.1-14 (SHEET 3 OF 7)

Net Free	Volume (ft³)	795.67	1572.59	402.0	142.3	348.52	145.14	324.23	668.07	460.28	788.23	671.14	315.67
Calc Peak Pressure	Differential (psiq)	8.25	8.25	13.08	13.04	12.47	12.0	11.41	11.43	12.24	12.63	10.87	10.69
	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>⊆</u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	110.51	223.84	89.33	31.62	89.14	32.94	83.81	176.1	102.29	197.63	176.84	81.91
	Height (ft)	8.0	8.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0
	Description	Over node 11; from el 187 to 195 ft.	Over node 13; from el 187 to 195 ft.	Around RCP 14; from el 195 to 200 ft.	Between RCP 4 and wall; from el 195 to 200 ft.	Between SG 4 and RCP 4; from el 195 to 200 ft.	Between SG 4 and HVAC shaft; from el 195 to 200 ft.	Between SG 4 and wall; from el 195 to 200 ft.	Between SG 4 and concrete beam; from el 195 to 200 ft.	Adjacent to node 32; from el 195 to 200 ft.	Opposite node 27; from el 195 to 200 ft.	Between SG 1 and concrete beam; from el 195 to 200 ft.	Between SG 1 and wall; from el 195 to 200 ft.
	Volume No.	23	24	25	26	27	28	59	30	31	32	33	8

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TABLE 6.2.1-14 (SHEET 4 OF 7)

Cross- Sectional Height Area (ft) (ft²) and 5.0 33.08	Cross-Sectional Area (ft²)	<u></u>		Initi Temperature (F) 120	Initial Conditions Pressure (psia) 13.2	Humidity (%)	Calc Peak Pressure Differential (psig) 10.22	Net Free Volume (ft³) 146.0
36	el 195 to 200 ft. Between SG 1 and RCP 1; from el 195 to 200 ft.	5.0	99.23	120	13.2	25	8.25	393.9
37	Between RCP 1 and wall; from el 195 to 200 ft.	5.0	37.59	120	13.2	25	8.23	169.15
38	Around RCP 1; from el 195 to 200 ft.	5.0	114.17	120	13.2	25	8.23	513.74
39	Opposite node 36; from el 195 to 200 ft.	5.0	225.29	120	13.2	25	8.60	912.67
40	Adjacent to node 39; from el 195 to 200 ft.	5.0	103.1	120	13.2	25	9.83	463.95
14	Between RCP 4 and northern wall of quadrant; from el 200 to 220 ft.	20.0	100.1	120	13.2	25	10.94	1796.2
42	Between RCP 4 and SW wall of quadrant; from el 200 to 220 ft.	20.0	35.3	120	13.2	25	10.9	635.37
43	Between HVAC shaft, RCP 4, and SG 4. from el 200 to 220 ft.	20.0	144.38	120	13.2	25	10.99	2577.68
44	Between SG 4 and southern wall of quadrant; from el 200 to 215 ft.	15.0	31.7	120	13.2	25	10.95	419.36
45	Between node 44 and quadrant 1; from el 200 to 207 ft.	7.0	111.91	102	13.2	25	10.93	96.269

TABLE 6.2.1-14 (SHEET 5 OF 7)

Net Free	Volume (ft³)	796.22	3015.61	1258.89	2214.86	3012.0	695.7	798.54	402.25	2640.6
Calc Peak Pressure	Differential (psiq)	10.74	10.95	11.07	11.0	10.36	10.58	10.46	10.32	8.21
	Humidity (%)	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>-</u>	Temperature (F)	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	110.59	225.12	93.84	214.07	224.85	111.55	110.91	30.43	147.88
	Height (ft)	8.0	15.0	15.0	20.0	15.0	7.0	8.0	15.0	20.0
	Description	Over node 45; from el 207 to 215 ft.	Between node 48 and quadrant 1; from el 200 to 215 ft.	Between SG 4 and northern wall, adjacent to node 49; from el 200 to 215 ft.	Adjacent to node 43; between RCP 4 and northern wall; from el 200 to 220 ft.	Between node 58 and quadrant 4; from el 200 to 215 ft.	Between node 53 and quadrant 4; from el 200 to 207 ft.	Over node 51; from el 207 to 215 ft.	Between SG 1 and southern wall of quadrant 1; from el 200 to 215 ft.	Between HVAC shaft, RCP 1, and SG 1; from el 200 to 220 ft.
	Volume No.	46	47	84	49	20	51	52	53	54

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TABLE 6.2.1-14 (SHEET 6 OF 7)

Net Free	Volume (ff³)	646.32	1986.14	2488.86	1347.85	918.0	596.4	404.2	1356.2	1234.0
Calc Peak Pressure	Differential (psiq)	7.98	8.0	8.26	8.95	4.01	4.60	3.59	4.13	4 12
	Humidity (%)	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
u	Temperature (F)	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft^2)	35.91	110.65	139.29	100.43	50.73	52.98	52.84	79.51	79.87
	Height (ft)	20.0	20.0	20.0	15.0	23.0	14.0	0.0	23.0	23.0
	Description	Between RCP 1 and SE wall of quadrant 1; from el 200 to 220 ft.	Between RCP 1 and northern wall of quadrant 1; from el 200 to 220 ft.	Adjacent to node 54; between RCP 1 and northern wall; from el 200 to 220 ft.	Between SG 1 and northern wall, adjacent to node 57; from el 200 to 215 ft.	NW node of left half of SG 4 doghouse; from el 215 to 238 ft.	SW node of left half of SG 4 dog- house; from el 215 to 229 ft.	Over node 60; from el 229 to 238 ft.	SE node of left half of SG 4 dog- house; from el 215 to 238 ft.	NE node of left half of SG 4 dog- house; from el 215 to 238 ft.
	Volume No.	55	26	57	28	26	09	19	62	63

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TABLE 6.2.1-14 (SHEET 7 OF 7)

Net Free Volume (ft³)	1234.0	1356.2	596.4	404.2	918.0	838.6	838.6	1231.6	1231.6	8.36E+04	2.75E+06
Calc Peak Pressure Differential (psiq)	4.0	4.01	4.33	3.51	3.86	10.82	7.63	8.27	5.96	3.41	3.10
Humidity (%)	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
In Temperature (F)	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional Area (ft²)	79.87	79.51	52.98	52.84	50.73		1	1	1	1	1
Height (ft)	23.0	23.0	14.0	0.6	23.0		1	1	1		ı
Description	Symmetric of node 63 in quadrant 1.	Symmetric of node 62 in quadrant 1.	Symmetric of node 60 in quadrant 1.	Symmetric of node 61 in quadrant 1.	Symmetric of node 59 in quadrant 1.	HVAC shaft 4	HVAC shaft 1.	HVAC duct 4.	HVAC duct 1.	Quadrants 2 and 3 of SG compartment.	Containment atmosphere.
Volume No.	64	65	99	29	89	69	70	7.1	72	73	74

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TABLE 6.2.1-15 (SHEET 1 OF 7)

STEAM GENERATOR COMPARTMENT MODEL (PIPE BREAK BELOW EL. 220 ft): NODE CHARACTERISTICS REACTOR COOLANT PUMP INLET NOZZLE (336-in.² BREAK)

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TABLE 6.2.1-15 (SHEET 2 OF 7)

Net Free	Volume (ft³)	1737.79	3729.58	2815.04	2400.44	766.38	892.25	887.09	796.98	1251.37	814.29	1581.52	1631.41
Calc Peak Pressure	Differential (psiq)	7.76	11.19	7.75	5.96	13.58	13.32	12.94	11.06	13.63	16.06	14.98	10.33
	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>:</u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	137.8	289.64	219.7	131.71	83.6	95.78	95.31	86.144	179.22	113.1	229.18	236.11
	Height (ft)	15.25	15.25	15.25	20.25	12.0	12.0	12.0	12.0	8.0	8.0	8.0	8.0
	Description	Between RCP 1, SG 1, and wall; from el 171 ft 9 in. to 187 ft.	Between hot, cold, and suction legs 1; from el 171 ft 9 in. to 187 ft.	Between hot leg 1 and wall; from el 171 fl 9 in. to 187 fl.	Interface of two halves of SG compartment between quadrants 1 and 2.	Over node 14; from el 183 to 195 ft.	Over node 5; from el 183 to 195 ft.	Over node 9 in quadrant 1; from el 183 to 195 ft.	Over node 10 in quadrant 1; from el 183 to 195 ft.	Over node 2; from el 187 to 195 ft.	Over node 3; from el 187 to 195 ft.	Over node 7; from el 187 to 195 ft.	Over node 12; from el 187 to 195 ft.
	Volume No.		12	5	4	15	16	17	18	19	20	21	22

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TABLE 6.2.1-15 (SHEET 3 OF 7)

Net Free	Volume (ft³)	795.67	1572.59	402.0	142.3	348.52	145.14	324.23	668.07	460.28	788.23	671.14	315.67
Calc Peak Pressure	Differential (psiq)	8.73	8.75	13.10	13.15	13.31	12.84	12.11	12.14	12.82	13.06	11.63	11.24
	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>c</u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft^2)	110.51	223.84	89.33	31.62	89.14	32.94	83.81	176.1	102.29	197.63	176.84	81.91
	Height (ft)	8.0	8.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	2.0
	Description	Over node 11; from el 187 to 195 ft.	Over node 13; from el 187 to 195 ft.	Around RCP 4; from el 195 to 200 ft.	Between RCP 4 and wall; from el 195 to 200 ft.	Between SG 4 and RCP 4; from el 195 to 200 ft.	Between SG 4 and HVAC shaft; from el 195 to 200 ft.	Between SG 4 and wall; from el 195 to 200 ft.	Between SG 4 and concrete beam; from el 195 to 200 ft.	Adjacent to node 32; from el 195 to 200 ft.	Opposite node 27; from el 195 to 200 ft.	Between SG 1 and concrete beam; from el 195 to 200 ft.	Between SG 1 and wall; from el 195 to 200 ft.
	Volume No.	23	24	25	26	27	28	59	30	31	32	33	8

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TABLE 6.2.1-15 (SHEET 4 OF 7)

Calc Peak Pressure	- I	10.75	8.75 393.9	8.83 169.15	8.76 513.74	9.13 912.67	10.46 463.95	11.48	11.42 635.37	11.56 2577.68	11.67 419.36	11.57 697.96
	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
_	Temperature (F)	120	120	120	120	120	120	120	120	120	120	102
Cross-	Area (ff²)	33.08	99.23	37.59	114.17	225.29	103.1	100.1	35.3	144.38	31.7	111.91
	Height (ft)	5.0	5.0	5.0	5.0	5.0	5.0	20.0	20.0	20.0	15.0	7.0
	Description	Between SG 1 and HVAC shaft; from el 195 to 200 ft.	Between SG 1 and RCP 1; from el 195 to 200 ft.	Between RCP 1 and wall; from el 195 to 200 ft.	Around RCP 1; from el 195 to 200 ft.	Opposite node 36; from el 195 to 200 ft.	Adjacent to node 39; from el 195 to 200 ft.	Between RCP 4 and northern wall of quadrant; from el 200 to 220 ft.	Between RCP 4 and SW wall of quadrant; from el 200 to 220 ft.	Between HVAC shaft, RCP 4, and SG 4. from el 200 to 220 ft.	Between SG 4 and southern wall of quadrant; from el 200 to 215 ft.	Between node 44 and quadrant 1;
	Volume No.	35	36	37	38	39	40	4	24	43	4	45

TABLE 6.2.1-15 (SHEET 5 OF 7)

Net Free	Volume (ft³)	796.22	3015.61	1258.89	2214.86	3012.0	695.7	798.54	402.25	2640.6
Calc Peak Pressure	Differential (psiq)	11.37	11.56	11.67	11.58	10.97	11.14	11.03	10.88	8.69
	Humidity (%)	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>c</u>	Temperature (F)	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	110.59	225.12	93.84	124.07	224.85	111.55	110.91	30.43	147.88
	Height (ft)	8.0	15.0	15.0	20.0	15.0	7.0	8.0	15.0	20.0
	Description	Over node 45; from el 207 to 215 ft.	Between node 48 and quadrant 1; from el 200 to 215 ft.	Between SG 4 and northern wall, adjacent to node 49; from el 200 to 215 ft.	Adjacent to node 43; between RCP 4 and northern wall; from el 200 to 220 ft.	Between node 58 and quadrant 4; from el 200 to 215 ft.	Between node 53 and quadrant 4; from el 200 to 207 ft.	Over node 51; from el 207 to 215 ft.	Between SG 1 and southern wall of quadrant 1; from el 200 to 215 ft.	Between HVAC shaft, RCP 1, and SG 1; from el 200 to 220 ft.
	Volume No.	46	47	84	64	20	51	52	53	54

TABLE 6.2.1-15 (SHEET 6 OF 7)

Net Free	Volume (ft³)	646.32	1986.14	2488.86	1347.85	918.0	596.4	1404.2	1356.2	1234.0
Calc Peak Pressure	Differential (psiq)	8.46	8.48	8.76	9.49	3.76	4.42	3.29	3.89	3.88
	Humidity (%)	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>'a</u>	Temperature (F)	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	35.91	110.65	139.29	100.43	50.73	52.98	52.84	79.51	79.87
	Height (ft)	20.0	20.0	20.0	15.0	23.0	14.0	0.6	23.0	23.0
	Description	Between RCP 1 and SE wall of quadrant 1; from el 200 to 220 ft.	Between RCP 1 and northern wall of quadrant 1; from el 200 to 220 ft.	Adjacent to node 54; between RCP 1 and northern wall; from el 200 to 220 ft.	Between SG 1 and northern wall, adjacent to node 57; from el 200 to 215 ft.	NW node of left half of SG 4 doghouse; from el 215 to 238 ft.	SW node of left half of SG 14 doghouse; from el 215 to 229 ft.	Over node 60; from el 229 to 238 ft.	SE node of left half of SG 14 dog- house; from el 215 to 238 ft.	NE node of left half of SG 4 doghouse; from el 215 to 238 ft.
	Volume No.	55	56	22	28	29	09	61	62	63

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TABLE 6.2.1-15 (SHEET 7 OF 7)

Net Free Volume (ft³)	1234.0	1356.2	596.14	404.2	918.0	838.6	838.6	1231.6	1231.6	8.36E+O14	2.75E+O6
Calc Peak Pressure Differential (psiq)	3.76	3.77	4.13	3.20	3.60	11.45	8.04	8.72	6.20	3.06	2.73
Humidity (%)	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
Temperature (F)	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional Area (ff²)	79.87	79.51	52.98	52.84	50.73		1	1	1	1	1
Height (ft)	23.0	23.0	14.0	0.6	23.0		1	1	1		
<u>Description</u>	Symmetric of node 63 in quadrant 1.	Symmetric of node 62 in quadrant 1.	Symmetric of node 60 in quadrant 1.	Symmetric of node 61 in quadrant 1.	Symmetric of node 59 in quadrant 1.	HVAC shaft 4	HVAC shaft 1.	HVAC duct 4.	HVAC duct 1.	Quadrants 2 and 3 of SG compartment.	Containment atmosphere.
Volume No.	64	65	99	29	89	69	20	7.1	72	73	74

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TABLE 6.2.1-16 (SHEET 1 OF 7)

STEAM GENERATOR COMPARTMENT MODEL (PIPE BREAK BELOW EL 220 ft): NODE CHARACTERISTICS STEAM GENERATOR INLET ELBOW (763-in.² BREAK)

Net Free	Volume (ft³)	1316.42	2283.14	1661.50	809.05	895.04	3124.98	3721.19	3078.62	890.21	837.79
Calc Peak Pressure	Differential (psiq)	10.54	19.63	20.52	20.97	21.34	22.13	22.16	20.93	21.12	17.50
	Humidity (%)	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>'</u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	222.97	176.0	132.24	83.6	95.78	165.73	189.03	163.51	95.31	86.44
	Height (ft)	6.56	15.25	15.25	11.25	11.25	23.25	15.25	23.25	11.25	11.25
	Description	Interface of two halves of SG compartment.	Between cold leg and wall; from el 171 ft 9 in. to 187 ft.	Between RCP 4, SG 4, and wall; from el 171 ft 9 in. to 187 ft.	By SG 4; from el 171 ft 9 in. to 183 ft.	Adjacent to SG 4; from el 171 ft 9 in. to 183 ft.	Around SG 4; from el 171 ft 9 in. to 195 ft.	Between hot, cold, and suction legs; from el 171 ft 9 in. to 187 ft.	Around SG 1; from el 171 ft 9 in. to 195 ft.	Adjacent to SG 1; from el 171 ft 9 in. to 183 ft.	By SG 1; from el 171 ft 9 in. to 183 ft.
	Volume No.	-	2	т	4	ഹ	9	_	∞	თ	10

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TABLE 6.2.1-16 (SHEET 2 OF 7)

Net Free Volume (ft³)	1737.79	3729.58	2815.04	2400.44	766.38	892.25	887.09	796.98	1251.37	814.29	1581.52	1631.41
Calc Peak Pressure Differential (psig)	12.69	18.17	12.66	9.83	20.90	21.22	21.03	17.52	19.66	20.54	22.03	16.74
Humidity (%)	25	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
Inii Temperature (F)	120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional Area (ft²)	137.8	289.64	219.7	131.71	83.6	95.78	95.31	86.44	179.22	113.1	229.18	236.11
Height (ft)	15.25	15.25	15.25	20.25	12.0	12.0	12.0	12.0	8.0	8.0	8.0	8.0
Description	Between RCP 1, SG 1, and wall; from el 171 ft 9 in. to 187 ft.	Between hot, cold, and suction legs 1; from el 171 ft 9 in. to 187 ft.	Between hot leg 1 and wall; from el 171 ft 9 in. to 187 ft.	Interface of two halves of SG compartment; between quadrants 1 and 2.	Over node 4; from el 183 to 195 ft.	Over node 5; from el 183 to 195 ft.	Over node 9 in quadrant 1; from el 183 to 195 ft.	Over node 10 in quadrant 1; from el 183 to 195 ft.	Over node 2; from el 187 to 195 ft.	Over node 3; from el 187 to 195 ft.	Over node 7; from el 187 to 195 ft.	Over node 12; from el 187 to 195 ft.
Volume No.		12	13	4	15	16	17	8	6	20	27	22

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TABLE 6.2.1-16 (SHEET 3 OF 7)

Net Free	Volume (ft³)	795.67	1572.59	402.0	142.3	348.52	145.14	324.23	668.07	460.28	788.23	671.4	315.67
Calc Peak Pressure	Differential (psiq)	14.13	14.21	19.22	19.26	19.28	19.24	18.83	18.93	19.31	19.34	18.44	17.86
	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	19.28	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>:</u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	110.51	223.84	89.33	31.62	89.14	32.94	83.81	176.1	102.29	197.63	176.84	81.91
	Height (ft)	8.0	8.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0
	Description	Over node 11; from el 187 to 195 ft.	Over node 13; from el 187 to 195 ft.	Around RCP 4; from el 195 to 200 ft.	Between RCP 4 and wall; from el 195 to 200 ft.	Between SG 4 and RCP 4; from el 195 to 200 ft.	Between SG 4 and HVAC shaft; from el 195 to 200 ft.	Between SG 4 and wall; from el 195 to 200 ft.	Between SG 4 and concrete beam; from el 195 to 200 ft.	Adjacent to node 32; from el 195 to 200 ft.	Opposite node 27; from el 195 to 200 fl. 200 fl.	Between SG 1 and concrete beam; from el 195 to 200 ft.	Between SG 1 and wall; from el 195 to 200 ft.
	Volume No.	23	24	25	26	27	28	59	30	31	32	33	8

TABLE 6.2.1-16 (SHEET 4 OF 7)

Net Free	Volume (ft³)	146.0	393.9	169.15	513.74	912.67	463.95	1796.2	635.37	2577.68	419.36	96.769
Calc Peak Pressure	Differential (psiq)	17.03	14.21	14.41	14.22	14.93	16.82	17.55	17.52	17.72	17.72	17.94
	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>=</u>	Temperature (F)	120	120	120	120	120	120	120	120	120	120	102
Cross- Sectional	Area (ft^2)	33.08	99.23	37.59	114.17	225.29	103.1	100.1	35.3	144.38	31.7	11.91
	Height (ft)	5.0	5.0	5.0	5.0	5.0	5.0	20.0	20.0	20.0	15.0	7.0
	Description	Between SG 1 and HVAC shaft; from el 195 to 200 ft.	Between SG 1 and RCP 1; from el 195 to 200 ft.	Between RCP 1 and wall; from el 195 to 200 ft.	Around RCP 1; from el 195 to 200 ft.	Opposite node 36; from el 195 to 200 ft.	Adjacent to node 39; from el 195 to 200 ft.	Between RCP 4 and northern wall of quadrant; from el 200 to 220 ft.	Between RCP 4 and SW wall of quadrant; from el 200 to 220 ft.	Between HVAC shaft, RCP 4, and SG 4; from el 200 to 220 ft.	Between SG 4 and southern wall of quadrant; from el 200 to 215 ft.	Between node 44 and quadrant 1; from el 200 to 207 ft.
	Volume No.	35	36	37	38	39	40	14	42	43	44	45

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TABLE 6.2.1-16 (SHEET 5 OF 7)

Net Free	Volume (ft³)	796.22	3015.61	1258.89	2214.86	3012.0	2.569	798.54	402.25	2640.6
Calc Peak Pressure	Differential (psiq)	17.63	17.82	17.82	17.73	17.28	17.71	17.38	17.20	14.08
	Humidity (%)	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
<u>-</u>	Temperature (F)	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ff²)	110.59	225.12	93.84	124.07	224.85	111.55	110.91	30.43	147.88
	Height (ft)	8.0	15.0	15.0	20.0	15.0	7.0	8.0	15.0	20.0
	Description	Over node 45; from el 207 to 215 ft.	Between node 48 and quadrant 1; from el 200 to 215 ft.	Between SG 4 and northern wall, adjacent to node 49; from el 200 to 215 ft.	Adjacent to node 43; between RCP 4 and northern wall; from el 200 to 220 ft.	Between node 58 and quadrant 4; from el 200 to 215 ft.	Between node 53 and quadrant 4; from el 200 to 207 ft.	Over node 51; from el 207 to 215 ft.	Between SG 1 and southern wall of quadrant 1; from el 200 to 215 ft.	Between HVAC shaff, RCP 1, and SG 1; from el 200 to 220 ft.
	Volume No.	46	47	8	49	20	51	52	23	54

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TABLE 6.2.1-16 (SHEET 6 OF 7)

Net Free	(#3)	646.32	1986.114	2488.86	1347.85	918.0	596.4	404.2	1356.2	1234.0
Calc Peak Pressure Differential	(bisd)	13.78	13.82	14.23	15.26	5.65	6.62	4.92	5.85	5.83
H	(%)	25	25	25	25	25	25	25	25	25
Initial Conditions Pressure	(psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
In	(F)	120	120	120	120	120	120	120	120	120
Cross- Sectional	(ff ²)	35.91	110.65	139.29	100.43	50.73	52.98	52.84	79.51	79.87
H G Di	()	20.0	20.0	20.0	15.0	23.0	0.40	0.6	23.0	23.0
	Description	Between RCP 1 and SE wall of quadrant 1; from el 200 to 220 ft.	Between RCP 1 and northern wall of quadrant 1; from el 200 to 220 ft.	Adjacent to node 54; between RCP 1 and northern wall; from el 200 to 220 ft.	Between SG 1 and northern wall, adjacent to node 57; from el 200 to 215 ft.	NW node of left half of SG 4 doghouse; from el 215 to 238 ft.	SW node of left half of SG 4 dog- house; from el 215 to 229 ft.	Over node 60; from el 229 to 238 ft.	SE node of left half of SG 4 dog- house; from el 215 to 238 ft.	NE node of left half of SG 4 dog- house; from el 215 to 238 ft.
Volume	No.	55	56	22	28	29	09	61	62	63

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TABLE 6.2.1-16 (SHEET 7 OF 7)

Net Free	Volume (ft³)	1234.0	1356.2	596.4	404.2	918.0	838.6	838.6	1231.6	1231.6	8.36E+04	2.75E+06
Calc Peak Pressure	Differential (psiq)	5.69	5.71	6.30	14.83	5.47	17.45	13.11	13.69	10.24	14.63	14.07
	Humidity (%)	25	25	25	25	25	25	25	25	25	25	25
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
드	Temperature (F)	120	120	120	120	120	120	120	120	120	120	120
Cross- Sectional	Area (ft²)	79.87	79.51	52.98	52.84	50.73	1	ı	ı	ı		1
	Height (ft)	23.0	23.0	14.0	0.6	23.0	ı	1	1	ı		1
	Description	Symmetric of node 63 in quadrant 1.	Symmetric of node 62 in quadrant 1.	Symmetric of node 60 in quadrant 1.	Symmetric of node 61 in quadrant 1.	Symmetric of node 59 in quadrant 1.	HVAC shaft 14	HVAC shaft 1.	HVAC duct 14.	HVAC duct 1.	Quadrants 2 and 3 of SG compartment.	Containment atmosphere.
	Volume No.	64	65	99	29	89	69	70	71	72	73	74

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TABLE 6.2.1-16A
STEAM GENERATOR COMPARTMENT MODEL (PIPE BREAK ABOVE EL 220 ft):
FLOW CHARACTERISTICS

L/A (ff ⁻¹)	0.329	0.234	0.097	0.070	0.286	0.139	0.777	0.211	0.099	0.637	0.199	0.089	0.131	0.059	0.186	0.302	0.155	0.256	0.219	0.139	0.054	0.168	0.126	0.198
Total (K _T)	1.41	1.41	1.41	1.41	1.35	3.24	1.41	1.16	1.02	1.41	1.16	1.02	1.18	3.24	1.4	1.38	14.	1.35	1.38	1.29	1.29	1.29	1.41	1.45
Contraction (K)	0.41	0.41	0.41	0.41	0.35	0.20	0.41	0.16	0.02	0.41	0.16	0.02	0.18	0.20	0.41	0.38	0.41	0.35	0.38	0.29	0.29	0.29	0.41	0.45
Expansion (K)	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_	_
Turning and Obstruction (K)	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
Friction (K)	0	0	0	0	0	2.04	0	0	0	0	0	0	0	2.04	0	0	0	0	0	0	0	0	0	0
Flow Area (ft²)	22.8	39.1	52.4	75.8	23.2	73.0	30.5	32.6	45.4	22.8	34.3	48.26	49.1	73.0	63.5	13.44	52.4	19.06	19.12	86.0	178.6	52.1	93.7	29.5
Description of Flow <u>Choked Unchoked</u>	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×
Node Number (From-To)	2-1	4-1	1-5	1-7	1-11	1-12	2-3	2-5	2-12	3-4	3-6	3-12	4-7	4-12	2-6	5-8	2-9	6-9	7-10	6-8	8-11	9-10	10-11	9-12
Vent <u>Path</u>	_	7	က	4	S	9	7	∞	တ	10	7	12	13	41	15	16	17	18	19	20	21	22	23	24

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TABLE 6.2.1-17 (SHEET 1 OF 5)

STEAM GENERATOR COMPARTMENT MODEL (PIPE BREAK BELOW EL. 220 ft): FLOW CHARACTERISTICS STEAM GENERATOR INLET ELBOW (763-in. ² BREAK)

-	(#-1)	0.096	0.079	0.295	0.295	0.122	0.151	0.151	0.0623	0.0623	0.136	0.134	0.0698	0.0698	0.297	0.297	0.103	0.103	0.192	0.192	0.0509	0.0509	0.1303	0.1303	0.1265	0.1265	0.1093	0.1093	0.08965	0.08965	0.153	0.153	0.0310	0.1071	0.115	0.086	0.163	0.1063	0.42	0.0798	0.0798	0.185	0.185
- - -	(K _T)	1.38	1.0	1.27	1.27	1.25	1.24	1.24	1.27	1.27	1.20	1.20	1.41	1.41	1.29	1.29	1.28	1.28	1.25	1.25	1.27	1.27	1.26	1.26	1.07	1.07	1.17	1.17	1.25	1.25	1.29	1.29	1.16	1.17	1.22	1.30	1.38	1.40	1.35	1.05	1.05	1.18	1.18
Control	(K)	0.38	0	0.27	0.27	0.25	0.24	0.24	0.27	0.27	0.20	0.20	0.41	0.41	0.29	0.29	0.28	0.28	0.25	0.25	0.27	0.27	0.26	0.26	0.07	0.07	0.17	0.17	0.25	0.25	0.29	0.29	0.16	0.17	0.22	0.30	0.38	0.40	0.35	0.05	0.05	0.18	0.18
	Expansion (K)	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	0.7	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	0.7	1.0	1.0
Turning and	(K)	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
.; ; ;	(K)	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
Flow	(ff)	20.25	92.08	29.58	29.58	92.1	63.12	63.12	94.0	125.8	64.65	80.0	47.57	47.57	58.72	58.72	63.65	63.65	36.93	36.93	129.19	129.19	70.58	70.58	99.53	99.53	81.3	81.3	113.9	113.9	94.2	94.2	247.93	76.37	68.31	92.65	13.7	44.86	50.6	80.58	80.58	28.52	28.52
Description of Flow	Unchoked	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	× :	×	× :	×	×	×	×	×	×	×	×	×:	×	×
Desci of F	Choked																																										
Node	(From-To)	1-74	1-73	20-19	23-24	2-1	3-2	11-13	7-2	13-12	21-19	22-24	3-7	11-12	20-21	23-22	3-4	11-10	20-15	23-18	9-2	12-8	21-6	22-8	4-5	10-9	15-16	18-17	2-6	න ් ර	16-6	17-8	8-9 9	2-9	16-17	13-14	24-14	14-73	14-74	19-25	24-38	19-26	24-37
, to //	Path	~	7	က	4	2	9	7	∞	တ	10	-	12	13	4	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	4	42

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TABLE 6.2.1-17 (SHEET 2 OF 5)

A/1	(# <u>)</u>	0.112	0.112	0.0794	0.0794	0.0389	0.0389	0.0778	0.0778	0 2903	0.2903	0.2396	0.2396	0.161	0.161	0.1378	0.1378	0.113	0.113	0.195	0.195	0.321	0.321	0.0711	0.0711	0.102	0.102	0.0482	0.0482	0.43 0.43	0.30	0.30	0.44	0.44	0.36	0.36	0.31	0.31	0.13	0.13	0.15	0.15 0.55	
Total	(K _T)	1.0	1.0	1.15	1.15	1.27	1.27	1.22	1.22	130	1.30	1.16	1.16	1.10	1.10	1.10	1.10	1.26	1.26	1.40	1.40	1.37	1.37	1.20	1.20	.18	1.18	1.20	02.1	1.7 4 7 4	1.39	1.39	1.74	1.74	1.28	1.28	1.20	1.20	1.10	1.10	1.34	1.34 1.56	
Contraction	(K)	0	0	0.15	0.15	0.27	0.27	0.22	0.22	0.30	0:30	0.16	0.16	0.10	0.10	0.10	0.10	0.26	0.26	0.40	0.40	0.37	0.37	0.20	0.20	0.18	0.18	0.20	0.20	0.74 4.70	0.39	0.39	0.74	0.74	0.28	0.28	0.20	0.20	0.10	0.10	0.34	0.34 0.56	
Expansion	(K)	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	10	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	0.0	0.6		- - 5	0.	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	0. C.	
Turning and Obstruction	(K)	0	0	0	0	0	0	0	0	C	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0 (0 0	> C	o c	0	0	0	0	0	0	0	0	0	0	0 (00	
Friction	(K)	0	0	0	0	0	0	0	0	C	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0 (> C	o c	0	0	0	0	0	0	0	0	0	0	0 (00	
Flow Area	(<u>₩</u>	53.45	53.45	62.58	62.58	81.5	81.5	57.19	57.19	12.58	12.58	21.43	21.43	66.88	66.88	76.62	76.62	80.58	80.58	11.0	11.0	6.89	6.89	107.33	107.33	85.83	85.83	1/4.0	1/4.0	18.42	34.77	34.77	17.66	17.66	29.7	29.7	26.72	26.72	68.93	68.93	58.55	58.55 13.75	
Description of Flow	Unchoked	×	×	×	×	×	×	×	×	×	:×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	× :	× :	× >	< >	< >	<×	×	×	×	×	×	×	×	×	×	× :	××	
Des	Choked																																										
Node Number	(From-To)	19-32	24-39	20-27	23-36	21-32	22-39	21-31	22-40	15-28	18-35	15-29	18-34	4-15	10-18	5-16	9-17	6-30	8-33	16-29	17-34	16-30	17-33	2-19	13-24	3-20	11-23	7-21	7Z-ZZ	38-37	25-32	38-39	26-27	37-36	27-32	36-39	27-28	36-35	32-31	39-40	31-30	40-33 28-29	
Vent	Path	43	44	45	46	47	48	49	20	51	52	53	54	22	56	22	28	29	09	61	62	63	2	92	99	29	89	9 69	? ?	- 62	73	74	75	9/	77	78	79	80	84	85	£ 33	2 8	

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TABLE 6.2.1-17 (SHEET 3 OF 5)

L/A (ff¹)	0.55	0.35	0.35	4.0	0.23	0.154	0.154	0.434	0.434	0.138	0.138	0.119	0.119	0.615	0.615	0.17	0.17	0.56	0.56	0.16	0.16	0.814	0.814	0.76	0.76	0.201	0.201	0.070	0.070	0.16	0.123	0.123	0.338	0.338	0.0794	0.0794	0.095	0.085	0.276	0.270	0.239	0.239	0.475	
Total $(K_{\overline{1}})$	1.56	1.23	1.23	1.38	1.35	1.21	1.21	1.21	1.21	1.22	1.22	1.25	1.25	1.29	1.29	1.13	1.13	1.10	1.10	1.13	1.13	1.27	1.27	1.18	1.18	1.09 90.1	1.09	- - -	5	1.10	1.26	1.26	1 .	4. 4.	1.47	7.47	1.43 6.43	4. 5.	1.46 9.4	- 4 5 6		25. 25.	1.35	
Contraction (K)	0.56	0.23	0.23	0.38	0.35	0.21	0.21	0.21	0.21	0.22	0.22	0.25	0.25	0.29	0.29	0.13	0.13	0.10	0.10	0.13	0.13	0.27	0.27	0.18	0.18	0.09	0.06	5.5	0.0	0.10	0.26	0.26	0.04	0.04	0.47	0.47	0.43	0.43 84.0	0.46	9.0	0.39	0.35	0.35	
Expansion (K)	1.0	1.0	1.0	1.0	0.7	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	0.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	0.0	0.6	0. c			0.	1.0	1.0	1.0	1.0	o. 4	0.5	o. 4		0. 6	- <i>4</i> 5 6	- t		<u>, </u>	
Turning and Obstruction (K)	0	0	4	0	0 (0 (0 (0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0 (0 (> C	o c	o c	0	0	0	0	0	0 0	o (0 0	> 0	> C	> 0	-	o c	0	
Friction (K)	0	0	0	0	0 (o (0 (0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0 (o (> C	o c	o c	0	0	0	0	0	0 0) (0 0	>	> C	>	-	o c	0	
Flow Area (ft²)	13.75	26.7	26.7	4.56	34.72	51.3	51.3	18.36	18.36	54.96	54.96	28.97	28.97	9.55	9.55	46.3	46.3	15.89	15.89	50.26	50.26	7.65	7.65	9.16	9.16	26.8	26.8 116.2	10.4	56.71	56.71	43.29	43.29	30.16	30.16	9.63	9.63	15.82	15.82		. E	21.33 24.55	 	8.63	
Description of Flow Unchoked	×	×	×	×	×	× :	× :	× :	×	× :	×	×:	×	×	×	×	×	×	×	×	×	×	×	×:	× :	× >	« >	< >	< ×	×														
Des of Choked																															×	×	×	×	××	< :	××	< >	« >	< >	< >	< ×	×	
Node Number (From-To)	35-34	29-30	34-33	29-34	30-33	25-41	38-56	26-42	37-25	27-43	36-54	32-49	39-57	32-43	39-24	32-48	39-28	31-48	40-58	31-47	40-50	28-43	35-54	28-44	35-53	30-45	33-51	33 50	29-45	34-51	41-74	56-74	42-74	55-74	43-74	54-74 1	49-74	50-74	50-64 47 63	20-74	52-65 46.62	53-66	44-60	
Vent Path	86	87	88	80	06	91	92	93	94	92	96	97	86	66	9	101	102	103	401	105	106	107	108	109	110	111	112	5 2	- <u>-</u> -	116	117	118	119	120	5 5	22.	123	4 5	5 5	0 7 7	72,	<u> </u>	13 6	

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TABLE 6.2.1-17 (SHEET 4 OF 5)

ΓA	(ff. ¹)	0.465	0.465	0.374	0.374	0.0828	0.0828	0.094	0.094	0.056	0.056	0.128	0.128	0.0893	0.0893	0.0482	0.0482	0.127	0.127	0.0406	0.0406	0.053	0.053	0.308	0.308	0.276	0.270	0.186	0.173	0.173	0.065	0.26	0.229	0.00	0.00	0.00	0.937	0.77	. 2.	1.2	0.41	0.41	0.55
Total	(K_T)	1.46	1.46	1.44	1.44	1.04	1.04	1.25	1.25	1.23	1.23	1.08	1.08	1.24	1.24	1.33	1.33	1.31	1.31	1.27	1.27	1.53	1.53	1.56	1.56	1.50		191	1.55	1.55	1.16	1.25	41.1	77.	13.5	5. 4. 5. 4.	. <u>.</u> .	- - -	0.	1.2	1.13	1.13	1.16
Contraction	3	0.46	0.46	4.0	0.44	0.04	0.04	0.25	0.25	0.23	0.23	0.08	0.08	0.24	0.24	0.33	0.33	0.31	0.31	0.27	0.27	0.53	0.53	0.56	0.56	0.50	0.00	0.61	0.55	0.55	0.16	0.25	0.14 7.00	0.27	0.27	90.0 30.0	0.00		<u>;</u> 0	0	0.13	0.13	0.16
Expansion	(X	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	0.5	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0			5 0	1.0	1.0	1.0	1.0		- -		5 5	5 6	5 6	<u>, (</u>	1.0	1.0	1.0	1.0
Turning and Obstruction	X	0	0	0	0	0	0	0	0	0	0	0	0	0	0 (o (0	0	0	0	0	0	0	0 (0 (o c	o c	o C	0	0	0	0 (> C	> C	o c	o c	o c	o c	0	0	0	0	0
Friction	(K)	0	0	0	0	0	0	0	0	0	0	0	0 (0	0 (Э,	0	0	0	0	0	0	0	0 (0 (o c	o c	o C	0	0	0	0 (o c	>	o c	o c	o c	o C	0	0	0	0	0
Flow Area	(# ₂)	10.05	10.05	9.95	9.95	91.35	91.35	81.18	81.18	128.07	128.07	78.89	78.89	107.19	107.19	85.73	85.73	95.48	95.48	109.22	109.22	178.11	178.11	22.12	22.12	29.81	23.01 27.88	27.88	53.57	53.57	175.22	35.04	58.39 21.0	0.1.0	13.0 13.5		16.38	16.38	10.53	10.53	0.59	30.59	23.0
Description of Flow	Unchoked	×				×	×	×	×	×	×	×	× :	× :	× :	× :	×:	×	×	×	×:	×:	×	× :	× :	× >	< >	<×	×	×	×	× >	« >	< >	< ×	< >	< ×	< ×	<×	:×	×	×	×
Des	Choked		×:	×:	×																																						
Node Number	(From-To)	54-66	43-60	28-68	48-59	45-46	51-52	41-42	56-55	41-49	26-57	42-43	55-54	43-49	54-57	43.48	54-58	43-44	54-53	49-58	57-58	48-47	28-20	44.45	53-51	44-46 53-52	20-00 77 47	51-50	46-47	52-50	47-50	45-51	46-52	00-60	59-61	20 00	09-04 90-62	66-65	61-62	67-65	62-63	65-64	63-29
Vent	Path	131	132	133	134	135	136	137	138	139	140	141	142	143	144	145	146	147	148	149	150	151	152	153	154	155 156	5 2 2 2 2 2	158	159	160	161	162	163	101	166	7 2 2	168	169	170	171	172	173	174

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TABLE 6.2.1-17 (SHEET 5 OF 5)

Y.	(ff ⁻¹)	0.55	0.276	0.276	0.11	0.11	0.215	0.215	0.236	0.236	0.181	0.181	1.256	1.256	1.273	1.273	0.0869	0.0869	0.776	0.776	0.98	0.98	0.795	0.795	0.008
Total	(K _T)	1.16	1.08	1.08	1.03	1.03	1.10	1.10	1.23	1.23	1.32	1.32	2.40	2.40	2.40	2.40	1.35	1.35	1.44	1.44	1.45	1.45	2.88	2.88	1.29
Contraction	(K)	0.16	0.08	0.08	0.03	0.03	0.10	0.10	0.23	0.23	0.32	0.32	0.50	0.50	0.50	0.50	0.35	0.35	0.44	0.44	0.44	0.44	0.45	0.45	0.29
Expansion	(K)	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0
Turning and Obstruction	到	0	0	0	0	0	0	0	0	0	0	0	06:0	06:0	06:0	06:0	0	0	0	0	0	0	1.38	1.38	0
Friction	(K)	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0.012	0.012	0.045	0.045	0
Flow Area	(ff ²)	23.0	38.1	38.1	42.0	42.0	47.25	47.25	29.4	29.4	15.54	15.54	21.6	21.6	5.4	5.4	75.6	75.6	7.2	7.2	38.88	33.88	39.26	39.26	391.01
Description of Flow	Unchoked	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×			×	×		×	×
Desc of F	Choked																		×	×			×		
Node	(From-To)	64-68	59-74	68-74	61-74	67-74	62-74	65-74	63-74	64-74	60-61	29-99	3-69	11-70	20-69	23-70	69-43	70-54	69-74	70-74	69-71	70-72	71-74	72-74	73-74
Vent	Path	175	176	177	178	179	180	181	182	183	184	185	186	187	188	189	190	191	192	193	194	195	196	197	198

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TABLE 6.2.1-18 (SHEET 1 OF 5)

STEAM GENERATOR COMPARTMENT MODEL (PIPE BREAK BELOW EL. 220 ft): FLOW CHARACTERISTICS STEAM GENERATOR INLET NOZZLE (306-in.² BREAK)

L/A (ff [*])	0.096 0.079 0.079 0.0523 0.0623 0.0623 0.0623 0.0698 0.0698 0.0509 0.050
Total $(K_{\overline{1}})$	\$0.2224442568444888888888255555555555555555
Contraction (K)	0. 00000000000000000000000000000000000
Expansion (K)	555555555555555555555555555555555555555
Turning and Obstruction (K)	000000000000000000000000000000000000000
Friction (K)	000000000000000000000000000000000000000
Flow Area (ft [*])	20,29,29,29,29,29,29,29,29,29,29,29,29,29,
Description of Flow Unchoked	**************************************
Des Of Choked	
Node Number (From-To)	
Vent Path	-uv4rorso51524557550022323333382828888888888444444444444444

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BLE 6.2.1-18	

L/A (ff. ¹)	0.000000000000000000000000000000000000
$\begin{array}{c} Total \\ (K_{\mathtt{T}}) \end{array}$	744686666666666666666666666666666666666
Contraction (K)	00000000000000000000000000000000000000
Expansion (K)	
Turning and Obstruction (K)	000000000000000000000000000000000000000
Friction (K)	000000000000000000000000000000000000000
Flow Arga (ff)	87774171717186876688811
Description of Flow Unchoked	**************************************
Des of <u>Choked</u>	
Node Number (From-To)	22224
Vent Path	\$\$40°0°0°0°0°0°0°0°0°0°0°0°0°0°0°0°0°0°0

IEET 3 OF 5)
6.2.1-18 (SH
TABLE (

L/A (ff ⁻¹)	0.000000000000000000000000000000000000
$\begin{array}{c} Total \\ (K_{T}) \end{array}$	### ### ##############################
Contraction (K)	00000000000000000000000000000000000000
Expansion (K)	
Turning and Obstruction (K)	000000000000000000000000000000000000000
Friction (K)	000000000000000000000000000000000000000
Flow Area (ff)	88.99.94.44.00.00.00.00.00.00.00.00.00.00.00.00
Description of Flow Unchoked	**************************************
Des <u>Choked</u>	
Node Number (From-To)	\$\$\$\$\$\$\$6.444\$\$6\$\$6\$\$6\$\$6\$\$6\$\$6\$\$6\$\$6\$\$6\$\$6\$\$6\$\$6\$
Vent Path	88000000000000000000000000000000000000

VEGP-FSAR-6	TABLE 6.2.1-18 (SHEET 4 OF 5)
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L/A (ff.1)	0.0406 0.0406 0.0553 0.
Total (K _T)	21111111111111111111111111111111111111
Contraction (K)	00000000000000000000000000000000000000
Expansion (K)	
Turning and Obstruction (K)	00000000000000000000000000000000000000
Friction (K)	0000 000000000000000000000000000000000
Flow Arga (ff)	8605777090772877882722225256600000000000000000000000000000
Description of Flow Unchoked	**************************************
Des of I <u>Choked</u>	
Node Number (From-To)	### ### ### ### ### ### ### ### ### ##
Vent Path	446626624686666666666666666666666666666

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TABLE 6.2.1-18 (SHEET 5 OF 5)

-	(#')	0.008
	(K _T)	1.29
ocitorita C	(K)	0.29
	(K)	1.0
Turning and	(K)	0
:: :::	(K)	0
Flow	(III)	391.01
Description of Flow	<u>Choked</u> <u>Unchoked</u>	×
Description of F	Choked	
Node	(From-To)	73-74
+u0//	Path	198

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TABLE 6.2.1-19 (SHEET 1 OF 5)

	L/A (ff')	0.0096 0.0096 0.0096 0.00623 0.00623 0.00623 0.00623 0.006
	$\begin{array}{c} Total \\ (K_{\mathtt{T}}) \end{array}$	\$-7-7-7-7-7-7-7-7-7-7-7-7-7-7-7-7-7-7-7
ERISTICS	Contraction (K)	0.000000000000000000000000000000000000
220 ft): FLOW CHARACTERISTICS i.* BREAK)	Expansion (K)	000000000000000000000000000000000000000
BREAK BELOW EL. 220 ft): ET NOZZLE (436-in. ² BREAK	Turning and Obstruction (K)	000000000000000000000000000000000000000
	Friction (K)	000000000000000000000000000000000000000
MPARTMENT MODEL STEAM GENERATOR	Flow Area (ff)	20.99.29 20.
STEAM GENERATOR CON	Description of Flow Unchoked	×× ××××××× ×× ××××××××××××××××××××××××
STEAM G	Description of FI	× × ×
	Node Number (From-To)	- 1-928-4-6-1-5-1-5-1-5-1-5-1-5-1-5-1-5-1-5-1-5-1
	Vent Path	-0w4vo/895174456/8550178488888888888888888888888

TABLE 6.2.1-19 (SHEET 2 OF 5)

L/A (ff ¹)	0.000000000000000000000000000000000000
Total $(K_{\overline{1}})$	7222886655555555555555555555555555555555
Contraction (K)	00000000000000000000000000000000000000
Expansion (K)	555555555555555555555555555555555555555
Turning and Obstruction (K)	000000000000000000000000000000000000000
Friction (K)	000000000000000000000000000000000000000
Flow Area (ff)	88888888821222222222222222222222222222
Description of Flow 1 Unchoked	**************************************
Desc of F Choked	
Node Number (From-To)	$\frac{92}{2}$
Vent Path	44¢¢¢¢¢¢¢¢¢¢¢86666666666666666666666666

TABLE 6.2.1-19 (SHEET 3 OF 5)

L/A (ff.1)	0.119 0.615 0.017 0.016
$\begin{array}{c} Total \\ \overline{(K_{\mathrm{T}})} \end{array}$	7,7,7,7,7,7,7,7,7,7,7,7,7,7,7,7,7,7,7,
Contraction (K)	00000000000000000000000000000000000000
Expansion (K)	000000000000000000000000000000000000000
Turning and Obstruction (K)	000000000000000000000000000000000000000
Friction (K)	000000000000000000000000000000000000000
Flow Area (ff²)	88.89.99.49.49.99.99.99.99.99.99.99.99.99.99
Description of Flow Unchoked	**************************************
Desc of F Choked	× × × × ××× ×
Node Number (From-To)	\$\\ \text{\te}\text{\tex
Vent Path	88 00 00 00 00 00 00 00 00 00 00 00 00 0

 $\frac{1}{12} \frac{1}{12} \frac$ Contraction $\begin{array}{c} 0.00\\$ Expansion (K) Turning and Obstruction (K) 0000 0000 0000 0000 0000 0000 TABLE 6.2.1-19 (SHEET 4 OF 5) Description of Flow Choked Node Number (From-To)

 $\frac{444}{800} + \frac{1}{12} + \frac{1}{1$

(H) A CONTROLL OF CONTROLL OF

TABLE 6.2.1-19 (SHEET 5 OF 5)

-	(# <u>'</u>)	0.008
H	(K _T)	1.29
1	Contraction (K)	0.29
	Expansion (K)	1.0
Turning and	(K)	0
	(K)	0
Flow	Area (ff)	391.01
Description of Flow	<u>Choked</u> <u>Unchoked</u>	×
Desc of F	Choked	
Node	(From-To)	73-74
7.77	vent Path	198

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TABLE 6.2.1-20 (SHEET 1 OF 5)

STEAM GENERATOR COMPARTMENT MODEL (PIPE BREAK BELOW EL. 220 ft): FLOW CHARACTERISTICS REACTOR COOLANT PUMP OUTLET NOZZLE (236-in.º BREAK)

L/A (ff¹)	0.096	0.295	0.295	0.122	0.151	0.151	0.0623	0.0623	0.136	0.134	0.0698	0.0698	0.297	0.297	0.103	0.103	0.192	0.192	0.0509	0.0509	0.1303	0.1303	0.1265	0.1265	0.1093	0.1093	0.08965	0.08965	0.153	0.153	0.0310	0.1071	0.115	0.086	0.163	0.1063	0.42	0.0798	0.0798	0.185	0.185
Total $(K_{\overline{1}})$	1.38	1.27	1.27	1.25	1.24	1.24	1.27	1.27	1.20	1.20	1.41	1.41	1.29	1.29	1.28	1.28	1.25	1.25	1.27	1.27	1.26	1.26	1.07	1.07	1.17	1.17	1.25	1.25	1.29	1.29	1.16	1.17	1.22	1.30	1.38	1.40	1.35	1.05	1.05	1.18	1.18
Contraction (K)	0.38	0.27	0.27	0.25	0.24	0.24	0.27	0.27	0.20	0.20	0.41	0.41	0.29	0.29	0.28	0.28	0.25	0.25	0.27	0.27	0.26	0.26	0.02	0.07	0.17	0.17	0.25	0.25	0.29	0.29	0.16	0.17	0.22	0.30	0.38	0.40	0.35	0.05	0.05	0.18	0.18
Expansion (K)	6. 4 0. 0	<u>, (</u>	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	0.5	0.7	0.7	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0
Turning and Obstruction (K)	00	0	0	0	0	0	0	0	0	0 (0 (0 (o (0 (0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
Friction (K)	00	0	0	0	0	0	0	0	0	0 (o (0 (o (0 (0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
Flow Area (ff ²)	20.25	97.08 29.58	29.58	92.1	63.12	63.12	94.0	125.8	64.65	80:0	47.57	47.57	58.72	58.72	63.65	63.65	36.93	36.93	129.19	129.19	70.58	70.58	99.53	99.53	81.3	81.3	113.9	113.9	94.2	94.2	247.93	76.37	68.31	92.65	13.7	44.86	9.09	80.58	80.58	28.52	28.52
Description of Flow 1 Unchoked	×>	<×	×	×	×	×	×	×	× :	× :	× :	× :	× :	× :	× :	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×
Desc of F <u>Choked</u>																																									
Node Number (From-To)	1-74	20-19	23-24	2-1	3-2	11-13	7-2	13-12	21-19	22-24	/-h	11-12	20-21	23-22	3.4	11-10	20-15	23-18	9-2	12-8	21-6	22-8	4-5	10-9	15-16	18-17	2-6	8-6 6	16-6	17-8	8-9	2-0	16-17	13-14	24-14	14-73	14-74	19-25	24-38	19-56	24-37
Vent Path	← 0	ν ω	4	2	9	7	ω	ග ්	9 :	, ,	7.5	, 3	4 ;	15	9	17	9	19	20	21	22	23	24	22	56	27	28	58	30	31	35	33	8	32	36	37	38	39	40	4	45

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TABLE 6.2.1-20 (SHEET 2 OF 5)

L/A (ff¹)	0.112 0.0794 0.0794 0.0389 0.0389 0.0778 0.2903 0.2396 0.1378 0.1378 0.1378 0.1378 0.1378 0.1378	0.0711 0.0711 0.102 0.0482 0.0482 0.30 0.30 0.30 0.31 0.31 0.31 0.31 0.31
Total $(\underline{K_{\overline{1}}})$	0.1.1.1.1.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.	0.21 0.21 0.21 0.21 0.21 0.21 0.21 0.21
Contraction (K)	0 0 0.15 0.22 0.22 0.30 0.22 0.10 0.10 0.10 0.37 0.37 0.37 0.37	0.20 0.20 0.18 0.20 0.20 0.20 0.28 0.20 0.39 0.10 0.34 0.34
Expansion (K)		
Turning and Obstruction (K)	000000000000000000000000000000000000000	000000000000000000000000000000000000000
Friction (K)	000000000000000000000000000000000000000	000000000000000000000000000000000000000
Flow Area (ff)	53.45 53.45 62.28 62.28 62.28 81.5 77.19 77.19 76.62 80.58 80.58 80.58 80.58 80.58 6.89 6.89 6.89	107.33 85.83 85.83 174.0 174.0 17.66 17.66 17.66 29.7 26.72 27.73
Description of Flow Unchoked	××××××××××××××××××××××××××××××××××××××	<>××××××××××××××××××××××××××××××××××××
Des of <u>Choked</u>		
Node Number (From-To)	19-32 20-27 20-27 22-39 22-39 22-39 15-28 15-28 16-39 16-39 16-39 17-34 16-39 17-34 16-30 17-34	2-16 3-24 11-23 11-23 12-22 25-26 38-37 27-38 36-37 36-35 36-35 36-35 36-35 36-35 36-35 36-35 36-35 36-35 36-35 36-35
Vent Path	£ 4 4 4 4 4 4 8 6 0 1 5 5 5 5 5 5 5 5 6 5 6 6 6 6 6 6 6 6 6	886888777747888888888888888888888888888

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TABLE 6.2.1-20 (SHEET 3 OF 5)

L/A (ff ⁻¹)	0.35	0.35 40.1	0.23	0.10 401.0	 4.434	0.434	0.138	0.138	0.119	0.615	0.615	0.17	0.17	0.56	0.56	0.16	0.16	0.814	0.76	0.76	0.201	0.201	0.076	0.16	0.16	0.123	0.338	0.338	0.0794	0.0794	0.095	0.035	0.276	0.259	0.259	0.475
Total $(K_{\overline{1}})$	1.23	1.38	1.35	- - - - - - - - - - - - - - - - - - -	<u> </u>	121	1.22	1.22	1.25	2.50	1.29	1.13	1.13	1.10	1.10		1.13	1.27	1.18	1.18	1.09		1.10	1.10	1.10	1.26	5 7 7	1.04	1.47	1.47	. 4 4. 4		- 1 9 1.4 1.4	1.39	ਦੇ ਹਵਾਲੇ ਹਵਾਲੇ	1.35
Contraction (K)	0.23	0.38 0.38	0.35	0.27	2.0	0.21	0.22	0.22	0.25	0.29	0.29	0.13	0.13	0.10	0.10	0.13	0.13	0.27	0.18	0.18	0.09	0.09	0.10	0.10	0.10	0.26	9.0	0.04	0.47	0.47	0.43	0.45 0.45	0.46	0.39	0.39 0.35	0.35
Expansion (K)	0.6	<u>5</u> 0.	0.7	o. c	5 6	0.1	1.0	1.0	0. 0. c	5.0	1.0	1.0	1.0	1.0	0.6	0.6	5. C	0.1	1.0	1.0	0.7		0.1	1.0	1.0	0. C	<u>, (</u>	1.0	1.0	1.0	<u>6</u> , 4	5.5	<u>, </u>	0.5		5 <u>6</u>
Turning and Obstruction (K)	0 0	00	0 (o c	o C	0	0	0	0 0	00	0	0	0	0	0 (o 0	o c	0	0	0	0 (-	0	0	0	o c	0	0	0	0	0 0	o c	0	0 (> C	00
Friction (K)	0 0	00	0 (> C	o C	0	0	0	0 0	o c	0	0	0	0	0 (> 0	o c	0	0	0	0 (-	0	0	0 (> C	0	0	0	0	0 0	o c	0	0 (-	00
Flow Area (ft²)	26.7	4.56	34.72	57.3 5.4.3	18.36	18.36	54.96	54.96	58.97	36.9/ 9.55	9.55	46.3	46.3	15.89	15.89	50.26 50.26	30.26 7.65	7.65	9.16	9.16	26.8	26.8 116.2	116.2	56.71	56.71	43.29 43.29	30.16	30.16	9.63	9.63	15.82	15.1	15.1	21.55	21.55 8.63	8.63
Description of Flow Unchoked	×>	<×	××	« »	<×	×	×	×	×>	<×	×	×	×	×	× >	« >	< ×	×	×	×	×	××	×	×	× :	× ×	<×	×	×	×	×>	< >	<×	×	« >	<×
Des of <u>Choked</u>																																				
Node Number (From-To)	29-30	29-33	30-33	25-41 38-56	26.42	37-55	27-43	36-54	32-49	32-57 32-43	39-54	32-48	39-28	31-48	40-58	31-47	28-43	35-54	28-44	35-53	30-45	33-51	33-50	29-45	34-51	41-/4 56-74	42-74	55-74	43-74	54-74	49-74	4 - 7 C	47-63	52-65	40-62 53.66	44-60
Vent Path	87	80 80	060	91 02	93 83	96	92	96	97	0 0	100	101	102	103	104	105	107	108	109	110	- 17	7 7 7	- - 1	115	116	11.7	110	120	121	122	123	10.4 40.4	126	127	128	130

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TABLE 6.2.1-20 (SHEET 4 OF 5)

L/A (ff')	0.465	0.374	0.0828	0.0828	0.094	0.094	0.056	0.030	0.128	0.0893	0.0893	0.0482	0.0482	0.127	0.12/	0.0406	0.053	0.053	0.308	0.308	0.276	0.276	0.186	0.173	0.173	0.065	0.229	09.0	09.0	0.937	0.937	0.77	0.77	<u>, t</u>	0.41	0.41	0.55
Total $(K_{\overline{1}})$	1.46 1.46	4. 4. 4.	10.1	1.04	1.25	1.25	1.23	2.53	1.08	1.24	1.24	1.33	ان ان د		127	127	1.53	1.53	1.56	1.56	05.1).51 161	1.61	1.55	1.55	1.16	- <u>-</u>	1.27	1.27	1.35	1.35		7.18 8 C	- -	5 <u>.</u> 1.	1.13	1.16
Contraction (K)	0.46 0.46	44.0		0.04	0.25	0.25	0.23	0.08	0.08	0.24	0.24	0.33	0.33	0.0	0.31	0.27	0.53	0.53	0.56	0.56	0.50	0.50	0.61	0.55	0.55	0.16 0.25	0.14	0.27	0.27	0.35	0.35	0.18	0.18	o c	0.13	0.13	0.16
Expansion (K)	0.0.	<u>6</u> 6	<u>, t</u>	1.0	1.0	0.6	0. 6	5 6	1.0	1.0	1.0	0.7		- - -		<u> </u>	0.	1.0	1.0	0.6	0. 6		1.0	1.0	0.7	0. 6	0.7	1.0	1.0	1.0	0.7	0. 7	o. 6	- - -	<u>5</u> 0	1.0	1.0
Turning and Obstruction (K)	00	00	0	0	0	0 (o c	o c	0	0	0	0 (> C	> C	o c	o C	0	0	0	0 (> 0	o c	0	0	0 (> C	0	0	0	0	0 (o (00	o c	0	0	0
Friction (K)	00	00	0	0	0	0 (0 0	o C	0	0	0	0 (o c	0 0	0 0	o C	0	0	0	0 (0 0	o c	0	0	0 (o c	0	0	0	0	0 (0 0	0 0	o c	0	0	0
Flow Area (ff²)	10.05 10.05	9.95	91.35	91.35	81.18	81.18	128.07	78.89	78.89	107.19	107.19	85.73	85.73	93.40	93.46 109.22	109.22	178.11	178.11	22.12	22.12	29.81	23.81 27.88	27.88	53.57	53.57	175.22 35.04	58.39	21.0	21.0	13.5	13.5	16.38	16.38	10.33	30,59	30.59	23.0
Description of Flow Unchoked	××	××	<×	×	×	××	×	<×	×	×	× :	× >	< >	< >	< ×	<×	×	×	×	× :	« >	< ×	×	×	×	× ×	<×	×	×	×	×	× >	×	< >	<×	×	×
Desc of I <u>Choked</u>																																					
Node Number (From-To)	54-66 43-60	58-68	45.46	51-52	41-42	56-55	41-49	42-43	55-54	43-49	54-57	43-48	54-58	44-64	24-33 49-58	57-58	48-47	58-50	44-45	53-51	44-46	53-52 45-47	51-50	46-47	52-50	47-50	46-52	29-60	99-89	59-61	68-67	60-62 60-67	66-65 61 63	01-02 67-65	62-63	65-64	63-59
Vent Path	131	133	135	136	137	138	139	<u> 4</u> 4 5 7 7 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	142	143	144	145	140 0 4	- 1	6 4 0 4 0 6	150	151	152	153	154	155 156	157	158	159	160	161	163	164	165	166	167	168	9 5 5	277	172	173	174

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TABLE 6.2.1-20 (SHEET 5 OF 5)

Υ /	(ff. ¹)	0.55	0.276	0.276	0.11	0.11	0.215	0.215	0.236	0.236	0.181	0.181	1.256	1.256	1.273	1.273	0.0869	0.0869	0.776	0.776	0.98	0.98	0.795	0.795	0.008
Total	(K _T)	1.16	1.08	1.08	1.03	1.03	1.10	1.10	1.23	1.23	1.32	1.32	2.40	2.40	2.40	2.40	1.35	1.35	1.44	1.44	1.45	1.45	2.88	2.88	1.29
Contraction	(K)	0.16	0.08	0.08	0.03	0.03	0.10	0.10	0.23	0.23	0.32	0.32	0.50	0.50	0.50	0.50	0.35	0.35	0.44	0.44	0.44	0.44	0.45	0.45	0.29
Expansion	(K)	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0
Turning and Obstruction	(K)	0	0	0	0	0	0	0	0	0	0	0	06.0	06:0	06.0	06:0	0	0	0	0	0	0	1.38	1.38	0
Friction	(K)	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0.012	0.012	0.045	0.045	0
Flow	(#3)	23.0	38.1	38.1	42.0	42.0	47.25	47.25	29.4	29.4	15.54	15.54	21.6	21.6	5.4	5.4	75.6	75.6	7.2	7.2	33.88	33.88	39.26	39.26	391.01
Description of Flow	Unchoked	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×
Descripti of Flow	Choked																								
Node	(From-To)	64-68	59-74	68-74	61-74	67-74	62-74	65-74	63-74	64-74	60-61	29-99	3-69	11-70	20-69	23-70	69-43	70-54	69-74	70-74	69-71	70-72	71-74	72-74	73-74
Vent	Path	175	176	177	178	179	180	181	182	183	184	185	186	187	188	189	190	191	192	193	194	195	196	197	198

TABLE 6.2.1-21 (SHEET 1 OF 5)

PE BREAK BELOW EL. 220 ft): FLOW CHARACTERISTICS IP INLET NOZZLE (336-in.² BREAK)	
STEAM GENERATOR COMPARTMENT MODEL (PIPE BREAK BELOW EL. 220 ft): FLOW CHARACTERISTI REACTOR COOLANT PUMP INLET NOZZLE (336-in.² BREAK)	

L/A (ff ⁻¹)	0.000000000000000000000000000000000000
Total (K ₁)	\$67774477766446888888888886677778886777788868868886677777
Contraction (K)	0 000000000000000000000000000000000000
Expansion (K)	50
Turning and Obstruction (K)	000000000000000000000000000000000000000
Friction (K)	000000000000000000000000000000000000000
Flow Area (ff)	0.0.000,000,000,000,000,000,000,000,000
Description of Flow Unchoked	************
Desc of F <u>Choked</u>	
Node Number (From-To)	-1-022-4-1-022-1-0
Vent Path	-uw4mor@o011024tbctse001024468788085888888888884444444444444444444444

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6.2.1-21 (S
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L/A (年)	00000000000000000000000000000000000000
Total (K _T)	7448866655588448888588888448884588855488888888
Contraction (K)	00000000000000000000000000000000000000
Expansion (K)	5
Turning and Obstruction (K)	000000000000000000000000000000000000000
Friction (K)	000000000000000000000000000000000000000
Flow Area (ff)	27771121268867688871168877788877778888888888
Description of Flow Unchoked X	·×××××××××××××××××××××××××××××××××××××
Des of <u>Choked</u>	
Node Number (From-To)	129~\$~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~
Vent Path 48	\$40°C'C'C'C'C'C'C'C'C'C'C'C'C'C'C'C'C'C'C'

-	(#.)	0.000000000000000000000000000000000000
F	(K _T)	72222222222222222222222222222222222222
	Contraction (K)	00000000000000000000000000000000000000
	Expansion (K)	
Turning and	Obstruction (K)	000000000000000000000000000000000000000
! : : : :	(K)	000000000000000000000000000000000000000
Flow	Area (ff)	800044477000000000000000000000000000000
Description of Flow	Unchoked	**************************************
Des	Choked	\times \times \times
Node	(From-To)	\$www.www.www.www.www.www.www.www.www.ww
7.77	vent Path	8890102246078901171141614161416141614161416141614161416

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۷/ ۱	(ff. ¹)	0.000.00000000000000000000000000000000
- - - -	(K _T)	
Contraction	(K)	00000000000000000000000000000000000000
TX Dansion	(K)	555555555555555555555555555555555555555
Turning and	(K)	0000 0000 0000000000000000000000000000
Friction	(X)	00000000000000000000000000000000000000
Flow	(# <u>)</u>	80000000000000000000000000000000000000
Description of Flow	Unchoked	**************************************
Desc of F	Choked	×
Node	(From-To)	### ### ### ### ### ### ### ### ### ##
Vent	Path	<u>4482236246862626646666666666666666666666</u>

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TABLE 6.2.1-21 (SHEET 5 OF 5)

<		0.008
Toto	(大) (大)	1.29
cottocrtac	(K)	0.29
	(K)	1.0
Turning and	(X)	0
riotio:	(X)	0
Flow	(£)	391.01
Description of Flow	Unchoked	×
Desc of F	Choked	
Node	(From-To)	73-74
, too//	Path	198

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STEAM GENERATOR COMPARTMENT MODEL (PIPE BREAK BELOW EL. 220 ft): FLOW CHARACTERISTICS LOOP CLOSURE WELD (336-in.² BREAK) TABLE 6.2.1-22 (SHEET 1 OF 5)

L/A	0.000000000000000000000000000000000000
Total	#0444444444444444444444444444444444444
Contraction	0 000000000000000000000000000000000000
Expansion	5 5555555555555555555555555555555555555
Turning and Obstruction	000000000000000000000000000000000000000
Friction	000000000000000000000000000000000000000
Flow Area	92.609.800.800.400.400.800.800.800.800.800.800
Description of Flow	
Desc of F	
Node Number	7-1-982-8-2-5-622-8-1-988-5-2-2-4-6-6-6-6-6-6-6-6-6-6-6-6-6-6-6-6-6
Vent	-0040000000000000000000000000000000000

VEGP-FSAR-6 TABLE 6.2.1-22 (SHEET 2 OF 5)

-	(# <u>·</u> 1	0.000000000000000000000000000000000000
F	(K ₁)	722686666666666666666666666666666666666
	Contraction (K)	00000000000000000000000000000000000000
	Expansion (K)	
Turning and	Obstruction (K)	000000000000000000000000000000000000000
i i i L	(K)	000000000000000000000000000000000000000
Flow	Area (ff)	### ### ### ### ### ### ### ### ### ##
Description of Flow	Unchoked	*****
Desci of F	Choked	
Node	(From-To)	922446444444444444444444444444444444444
*****	Vent	### ### ### ### ### ### ### ### ### ##

TABLE 6.2.1-22 (SHEET 3 OF 5)

VEGP-FSAR-6

Ϋ́	(#_)	0.000000000000000000000000000000000000
Total	(K _Τ)	2000 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
Confraction	(K)	0.000000000000000000000000000000000000
Expansion	(K)	
Turning and Obstruction	(K)	000000000000000000000000000000000000000
Friction	(K)	000000000000000000000000000000000000000
Flow Arga	(#)	88.9.9.9.4.4.6.6.0.0.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2
Description of Flow	Unchoked	**************************************
Des	Choked	\times \times \times
Node Number	(From-To)	%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
Vent	Path	886555546555656555757575757575757575757575

VEGP-FSAR-6 TABLE 6.2.1-22 (SHEET 4 OF 5)

-	(# <u>·</u> 1)	0.0000 0.00406 0.00406 0.00533 0.00653 0.00653 0.00653 0.00653 0.00653 0.00669
- 	(K ⊒	21111111111111111111111111111111111111
cito cito	Contraction (K)	00000000000000000000000000000000000000
	(K)	000000000000000000000000000000000000000
Turning and	Obstruction (K)	00000000000000000000000000000000000000
;;;;;;;;;;;;;;;;;;;;;;;;;;;;;;;;;;;;;;	(K)	00000000000000000000000000000000000000
Flow	Area (ff ²)	86951762922222222222222222222222222222222222
Description of Flow	Unchoked	**************************************
Des of F	Choked	×
Node	(From-To)	497-88848-4848-48-48-48-48-88-88-88-88-88-8
100/	Path	44626644666666666666666666666666666666

TABLE 6.2.1-22 (SHEET 5 OF 5)

4/1	(# <u>.</u> 1)	0.008
Total	(K _T)	1.29
Contraction	(K)	0.29
Expansion	(K)	1.0
Turning and Obstruction	(K)	0
Friction	(K)	0
Flow	(ff)	391.01
Description of Flow	Unchoked	×
Desc of Fl	Choked	
Node	(From-To)	73-74
Vent	Path	198

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TABLE 6.2.1-23 (SHEET 1 OF 2)

PRESSURIZER COMPARTMENT MODEL: FLOW CHARACTERISTICS (SPRAY LINE) LOSS COEFFICIENT (K)

L/A (ff ⁻¹)	0.027	0.190	0.157	0.60	0	00.	0.1	 	0.103	0.0	0.100	0.332	0.753	0.129	0.231	0.117	0.303	0.213	0.517	0.335	0.150	0.241	0.154	0.219	0.140	0.225	0.350	0.248	0.449	0.178	0.255	0.163	0.410	0.161	0.100	0.178	0.330	0.186	0.143	0.170	0.152	0.154
Total	1.82	1.23	1.32	6,7	 	0 7.	25.1 22.1	2.6	8. t	6.6	0 0	5.5	25.	25.	135	1.29	1.35	1.38	2.32	1.35	1.32	1.32	1.32	1.32	1.32	1.21	1.35	1.26	1.21	1.21	1.32	1.21	1.29	1.35	1.26	1.35	1.32	1.32	1.32	1.32	1.29	1.32
Contraction	0.82	0.23	0.32	0.20	92.0	0.20	0.32	0.20	0.30	0.23	0.30 35	0.00	20.0	0.32	0.35	0.29	0.35	0.38	0.26	0.35	0.32	0.32	0.32	0.32	0.32	0.21	0.35	0.26	0.21	0.21	0.32	0.21	0.29	0.35	0.26	0.35	0.32	0.32	0.32	0.32	0.29	0.32
Expansion	0.1	0.5	0. 6	5.6	5.6	5. 6	5. 6	5. 6	5 6		5 5		5 C	5 6	0 7	0.	0.0	0.	1.06	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	0.7	1.0	0.7	1.0
and Obstruction	0	o (00	o c	o c	-	o c	o c	o c	o c	> C	o c	o c	o C	o C	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
Turning Friction (ft²-h)	0	o (00	o c	o c	-	>	o c	o c	o c	o c	o c	o c	o C	o C	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
Area <u>(ft²)</u>	172.0	27.4	78.1 23.6	25.0	25.0	70.0 70.0	20.7 33.4	t. 14	27.72 7.8.4	t 4	20.3 12.7	30.7	20.0	33.0	19.9	35.0	16.1	14.0	12.7	13.5	25.2	19.2	27.8	22.9	29.4	27.9	9.5	24.9	15.3	32.5	16.5	37.3	17.8	14.5	51.0	10.5	23.0	24.2	29.3	26.0	31.5	33.0
Description Flow of Flow Choked Unchoked	*:	× :	× >	<>	<>	<>	<>	<>	< ×	<>	< >	<>	< >	(×	:×	: ×	: ×	: ×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×:	×
Node 5 (From-To)	1-28	2-1	2 2.3 4.3		† -	- 1 - 4	ֆ դ	- c	2-C	7-0	- C - C	2 0	2 4	- o		9-6	10-6	10-11	10-28	11-7	11-12	12-8	12-13	13-9	13-10	14-10	14-15	15-11	15-16	16-12	16-17	17-13	17-14	18-14	18-19	19-15	19-20	20-16	20-21	21-17	21-18	22-18
Vent <u>Parameters</u>	- (7 .	m <	† u	ວ ຜ	۸ ٥	~ α	o c	_∞	2 7	- 5	<u>4</u> ¢	5 4	ά	, C	17	. 6	9	20	21	22	23	24	25	56	27	78	59	30	31	32	33	8	32	36	37	38	36	40	14	45	43

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TABLE 6.2.1-23 (SHEET 2 OF 2)

	ΓĄ	(ff ⁻¹)	660.0	12.5	0.167	0.162	0.238	0.141	0.218	0.149	2.08	0.099	0.104	0.140	0.130	0.043	0.532	0.024
		Total	1.26	1.49	1.32	1.32	1.32	1.32	1.32	1.29	1.49	1.35	1.32	1.35	1.35	1.26	1.49	1.23
		Contraction	0.26	0.49	0.32	0.32	0.32	0.32	0.32	0.29	0.49	0.35	0.32	0.35	0.35	0.26	0.49	0.23
		Expansion	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0
Turning	and	Obstruction	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
	Friction	(ff²-h)	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
Flow	Area	Œ	51.8	0.20	29.2	27.5	19.2	29.7	24.9	32.0	1.23	26.1	27.0	13.4	16.4	161.5	5.62	304.3
Description	of Flow	<u>Choked</u> <u>Unchoked</u>	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×	×
	Node 5	(From-To)	22-23	22-28	23-19	23-24	24-20	24-25	25-21	25-22	25-28	26-22	26-23	26-24	26-25	26-27	26-28	27-28
	Vent	<u>Parameters</u>	4	45	46	47	48	49	20	51	52	53	54	22	26	22	28	59

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TABLE 6.2.1-24 (SHEET 1 OF 2)

PRESSURIZER COMPARTMENT FLOW CHARACTERISTIC MODE (SURGE LINE) LOSS COEFFICIENT (K)

L/A (ft ⁻¹)	0.027	0.157	0.201	0.114	0.153	0.119	0.143	0.105	0.319	0.180	0.332	0.753	0.129	0.231	0.117	0.303	0.213	0.517	0.335	0.150	0.241	0.0	0.410	0.225	0.350	0.248	0.449	0.178	0.255	0.163	0.410	0.161	0.100	0.178	0.330	0.186	0.143	0.170
Total	1.82	1.32	1.29	1.29	1.26	1.32	1.23	1.38	1.29	85.7		32.	1.32	1.35	1.29	1.35	1.38	2.32	1.35	1.32	1.32	25.	3.52	1.21	1.35	1.26	1.21	2.21	1.32	1.21	1.29	1.35	1.26	1.35	1.32	1.32	7.32	1.32
Contraction	0.82	0.32	0.29	0.29	0.26	0.32	0.23	0.38	0.29	0.38	0.33	0.32	0.32	0.35	0.29	0.35	0.38	0.26	0.35	0.32	0.32	0.32	0.32	0.21	0.35	0.26	0.21	0.21	0.32	0.21	0.29	0.35	0.26	0.35	0.32	0.32	0.32	0.32
Expansion	0.7	0.1	1.0	1.0	1.0	1.0	0.7	0.0	0.7	0. 6	- - -	5 C	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0 0. (0. 6	5.6	5 6	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	0. d	0.5	0. 0	0.0	0. C.
Turning and Obstruction	00	0	0	0	0	0	0	0 (0 0	> C	o c	o C	0	0	0	0	0	1.06	0	0 (o c	> C	o c	0	0	0	0	0	0	0	0	0	0	0 (0	0 0	- (00
Friction (ff²-h)	00	0	0	0	0	0	0	0 (0 (> C	o c	0 0	0	0	0	0	0	0	0	0 (5 C	-	o c	0	0	0	0	0	0	0	0	0	0	0 (0	0 (> (00
Flow Area <u>(ff²)</u>	172.0	18.1	23.6	25.0	26.8	26.2	33.4	27.5	16.4	20.5 43.4	30.7	20.00	33.0	19.9	35.0	16.1	14.0	12.7	13.5	25.2	19.2 27.0	0.72	29.4 29.4	27.9	9.5	24.9	15.3	32.5	16.5	37.3	17.8	14.5	51.0	10.5	23.0	24.2	29.3	26.0 31.5
Description of Flow Choked <u>Unchoked</u>	××	×	×	×	×	×	×	×	××	« >	< >	<×	×	×	×	×		×	×	×	« >	<>	<×	×	×	×	×	×	×	×	×	×	×	××	× :	××	<>	××
Node 5 (From-To)	1-28 2-1	2-3	3-1	3-4	1-4	4-5	5-1	5-2	6-2 0	/-Q	۲- د - ۲-	- 8 5 4	o-8	9-2	9-6	10-6	10-11	10-24	11-7	11-12	12-8	12.0	13-10	14-10	14-15	15-11	15-16	16-12	16-17	17-13	17-14	18-14	18-19	19-15	19-20	20-16	77-07	21-1 / 21-18
Vent <u>Parameters</u>	- 0	၊ က	4	വ	9	7	ω .	თ (9 7	- 5	<u>م</u> 5	5 4	. 15	16	17	18	19	20	21	52	8 8	45	8 8	27	78	59	30	31	32	33	84	32	36	37	88	30	04 6	41 42

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TABLE 6.2.1-24 (SHEET 2 OF 2)

Γ\A	(ft ⁻¹)	0.154	0.099	12.5	0.167	0.162	0.238	0.141	0.218	0.149	2.08	0.099	0.104	0.140	0.130	0.043	0.532	0.024
	Total	1.32	1.26	1.49	1.32	1.32	1.32	1.32	1.32	1.29	1.49	1.35	1.32	1.35	1.35	1.26	1.49	1.23
	Contraction	0.32	0.26	0.49	0.32	0.32	0.32	0.32	0.32	0.29	0.49	0.35	0.32	0.35	0.35	0.26	0.49	0.23
	Expansion	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0
Turning and	Obstruction	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
Friction	(ft²-h)	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
Flow Area	(ff ²)	33.0	51.8	0.2	29.2	27.5	19.2	29.7	24.9	32.0	1.23	26.1	27.0	13.4	16.4	161.5	5.62	304.3
Description of Flow	ed <u>Unchoked</u>	×	×		×	×	×	×	×	×		×	×	×	×			×
Des	Chok			×							×					×	×	
Node 5	(From-To)	22-18	22-23	22-28	23-19	23-24	24-20	24-25	25-21	25-22	25-28	26-22	26-23	26-24	26-25	26-27	26-28	27-28
Vent	<u>Parameters</u>	43	44	45	46	47	48	49	20	51	52	53	54	22	26	22	28	59

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TABLE 6.2.1-25 (SHEET 1 OF 4)
PRESSURIZER COMPARTMENT MODEL: NODE CHARACTERISTICS

Node 1

7

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4

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Calc Peak Pressure Idity Differential (psi)	3.15	5.71	5.84	0 5.70	5.64	7.41	0 7.46
Humidity (%)	50.0	50.0	50.0	50.0	50.0	50.0	50.0
I Conditions Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2
Initial Conditions Temperature Pressure (°F)	120	120	120	120	120	120	120
Net Volume (ff)	2596	400	367	485	524	350	317
Cross- Sectional Area (ff)	215.7	49.3	46.3	58.1	62.0	49.3	46.3
Height (ft)	12.67	10.33	10.33	10.33	10.33	11.0	11.0
Description	In pressurizer compartment between el 252 ft 4 in. and 265 ft.	Northwest quadrant in pressurizer com- partment between el 242 ft 0 in. and 252 ft 4 in.	Northeast quadrant in pressurizer compartment between el 242 ft 0 in. and 252 ft 4 in.	Southeast quadrant in pressurizer compartment between el 242 ft 0 in. and 252 ft 4 in.	Southwest quadrant in pressurizer compartment between el 242 ft 0 in. and 252 ft 4 in.	Northwest quadrant in pressurizer com- partment between el 231 ft 0 in. and 242 ft 0 in.	Northeast quadrant in pressurizer com- partment between el 231 ft 0 in. and 242 ft 0 in.

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TABLE 6.2.1-25 (SHEET 2 OF 4)

Node

Calc Peak Pressure	Unrerential (psi)	7.36	7.44	10.5	10.7	10.7	10.6	6	11.7
	HUMIGITY (%)	50.0	50.0	50.0	50.0	50.0	50.0	50.0	50.0
Initial Conditions	Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
Initial	emperature (°F)	120	120	120	120	120	120	120	120
Net	volume (ft ³)	144	480	296	269	374	409	180	163
Cross- Sectional	Area (ft)	58.1	62.0	49.3	46.3	58.1	62.0	49.3	46.3
4 2 	Height (ff)	11.0	11.0	9.34	9.34	9.34	9.34	5.66	5.66
	Description	Southeast quadrant in pressurizer com- partment between el 231 ft 0 in. and 242 ft 0 in.	Southwest quadrant in pressurizer com- partment between el 231 ft 0 in. and 242 ft 0 in.	Northwest quadrant of pressurizer compartment between el 221 ft 7 7/8 in. and 231 ft 0 in.	Northeast quadrant of pressurizer compartment between el 221 ft 7 7/8 in. and 231 ft 0 in.	Southeast quadrant of pressurizer compartment between el 221 ft 7 7/8 in. and 231 ft 0 in.	Southwest quadrant of pressurizer compartment between el 221 ft 7 7/8 in. and 231 ft 0 in.	Northwest quadrant of pressurizer com- partment between el 216 ft 0 in. and 221 ft 7 7/8 in.	Northeast quadrant of pressurizer com- partment between el 221 ft 0 in, and 221 ft 7 7/8 in.

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TABLE 6.2.1-25 (SHEET 3 OF 4)

Node

Calc Peak Pressure Differential (psi)	11.9	11.9	15.8	15.8	15.4	15.4	17.3	17.2
Humidity (%)	50.0	50.0	50.0	50.0	50.0	50.0	50.0	50.0
Initial Conditions ure Pressure (psia)	13.2	13.2	13.2	13.2	13.2	13.2	13.2	13.2
Initial Temperature (°F)	120	120	120	120	120	120	120	120
Net Volume (ff ³)	227	244	622	573	401	438	633	584
Cross- Sectional Area (ft)	58.1	62.0	81.3	76.2	58.1	62.0	81.3	76.2
Height (ft)	5.66	5.66	10.0	10.0	10.0	10.0	10.19	10.19
Description	Southeast quadrant of pressurizer compartment between el 216 ft 0 in. and 221 ft 7 7/8 in.	Southwest quadrant of pressurizer compartment between el 216 ft 0 in. and 221 ft 7 7/8 in.	Northwest quadrant in pressurizer com- partment between el 206 ft 0 in. and 216 ft 0 in.	Northeast quadrant in pressurizer com- partment between el 206 ft 0 in. and 216 ft 0 in.	Southeast quadrant in pressurizer compartment between el 206 ft 0 in. and 216 ft 0 in.	Southwest quadrant in pressurizer compartment between el 206 ft 0 in. and 216 ft 0 in.	Northwest quadrant in pressurizer com- partment between el 195 ft 9 3/4 in. and 206 ft 0 in.	Northeast quadrant in pressurizer com- partment between el 195 ft 9 3/4 in. and 206 ft 0 in.

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TABLE 6.2.1-25 (SHEET 4 OF 4)

Calc Peak Pressure Differential (psi)	17.1	17.2	20.7	6.51	ı
Humidity (%)	50.0	50.0	50.0	50.0	50.0
Conditions Pressure (psia)	13.2	13.2	13.2	13.2	13.2
Initial Conditions Temperature Pressure Humidity (°F) (Psia) (%)	120	120	120	120	120
Net Volume (ff)	409	445	2419	3496	2.75E+06
Cross- Sectional Area (ft)	1.85	62.0	277.7	277.7	I
Height (ft)	10.19	10.19	10.81	13.25	ı
Description	Southeast quadrant in pressurizer com- partment between el 195 ft 9 3/4 in. and 206 ft 0 in.	Southwest quadrant in pressurizer compartment between el 195 ft 9 3/4 in. and 306 ft 0 in.	Inside pressurizer compartment between el 185 ft 0 in. and 195 ft 9 3/4 in.	Inside pressurizer compartment between el 171 ft 9 in. and 185 ft 0 in.	Containment building free volume.
Node	24	25	56	27	28

a. Note that this value includes allowances for volumes occupied by large equipment as well as a 5 percent reduction to account for smaller objects.

b. Results are for a 308-in. surge line break except nodes 1 through 5, which are for a spray line break at the pressurizer nozzle.

TABLE 6.2.1-26 (SHEET 1 OF 7)

REACTOR CAVITY TIME DEPENDENT FLOW CONDITIONS

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
0.00000, 0.00250,	0.0, 1.3586147E+4,	0., 7.6261951E+6,
0.00500,	1.7940995E+4,	1.0071084E+7,
0.00752,	2.0648819E+4,	1.1591079E+7,
0.01001,	2.2913458E+4,	1.2857206E+7,
0.01251,	2.4062034E+4,	1.3493659E+7,
0.01501,	2.3856512E+4,	1.3362610E+7,
0.01751,	2.6598904E+4,	1.4911117E+7,
0.02003,	2.7096058E+4,	1.5177837E+7,
0.02253,	2.6427985E+4,	1.4785699E+7,
0.02507,	2.6171937E+4,	1.4631801E+7,
0.02756,	2.6566208E+4,	1.4848680E+7,
0.03005,	2.6676953E+4,	1.4905002E+7,
0.03252,	2.7098266E+4,	1.5139974E+7,
0.03510,	2.7742642E+4,	1.5502190E+7,
0.03751,	2.8169787E+4,	1.5741567E+7, 1.5968826E+7,
0.04005, 0.04257,	2.8575125E+4, 2.8940040E+4,	1.6173308E+7,
0.04207,	2.9015063E+4,	1.6212095E+7,
0.04753,	2.8845152E+4,	1.6111615E+7,
0.05000,	2.8677447E+4,	1.6013188E+7,
0.05251,	2.8523145E+4,	1.5923076E+7,
0.05510,	2.8416210E+4,	1.5860480E+7,
0.05752,	2.8345423E+4,	1.5818843E+7,
0.06003,	2.8188576E+4,	1.5728214E+7,
0.06257,	2.7840963E+4,	1.5529144E+7,
0.06505,	2.7390292E+4,	1.5271875E+7,
0.06759,	2.7024295E+4,	1.5063533E+7,
0.07005,	2.6934059E+4,	1.5012643E+7,
0.07261,	2.7117701E+4,	1.5117598E+7,
0.07501,	2.7372228E+4,	1.5262759E+7,
0.07759,	2.7520339E+4,	1.5346859E+7,
0.08005,	2.7439861E+4,	1.5300528E+7,
0.08255,	2.7153041E+4,	1.5136868E+7,
0.08502,	2.6724728E+4,	1.4893012E+7,
0.08761,	2.6197717E+4,	1.4593675E+7,
0.09002,	2.5681851E+4,	1.4301168E+7,
0.09253,	2.5203708E+4,	1.4030333E+7,
0.09502,	2.4835535E+4,	1.3822267E+7,
0.09754,	2.4578990E+4,	1.3677537E+7,
0.10013,	2.4419910E+4,	1.3588081E+7,
0.10253,	2.4355729E+4,	1.3552374E+7,

TABLE 6.2.1-26 (SHEET 2 OF 7)

	Mass Flow	Energy Flow
Time (s)	(lb/s)	(Btu/s)
0.10500,	2.4365128E+4,	1.3558416E+7,
0.10751,	2.4455309E+4,	1.3610180E+7,
0.11010,	2.4625670E+4,	1.3707282E+7,
0.11255,	2.4845949E+4,	1.3832514E+7,
0.11503,	2.5086307E+4,	1.3969057E+7,
0.12010,	2.5466453E+4,	1.4184861E+7,
0.12260,	2.5551384E+4,	1.4232951E+7,
0.12513,	2.5567152E+4,	1.4241650E+7,
0.12761,	2.5523282E+4,	1.4216391E+7,
0.13014,	2.5427294E+4,	1.4161608E+7,
0.13265,	2.5276425E+4,	1.4075754E+7,
0.13507,	2.5071667E+4,	1.3959456E+7,
0.13754,	2.4813584E+4,	1.3813099E+7,
0.14005,	2.4539029E+4,	1.3657649E+7,
0.14258,	2.4313941E+4,	1.3530409E+7,
0.14509,	2.4165958E+4,	1.3446888E+7,
0.14765,	2.4092923E+4,	1.3405844E+7,
0.15005,	2.4058579E+4,	1.3386724E+7,
0.15256,	2.4033247E+4,	1.3372693E+7,
0.15502,	2.3993516E+4,	1.3350446E+7,
0.15762,	2.3909192E+4,	1.3302862E+7,
0.16014,	2.3778645E+4,	1.3229134E+7,
0.16258,	2.3624748E+4,	1.3142275E+7,
0.16515,	2.3480926E+4,	1.3061202E+7,
0.16764,	2.3386045E+4,	1.3007825E+7,
0.17012,	2.3341080E+4,	1.2982715E+7,
0.17261,	2.3337016E+4,	1.2980780E+7,
0.17506,	2.3360386E+4,	1.2994312E+7,
0.17760,	2.3404110E+4,	1.3019393E+7,
0.18014,	2.3465021E+4,	1.3054108E+7,
0.18255,	2.3537716E+4,	1.3095460E+7,
0.18503,	2.3632725E+4,	1.3149368E+7,
0.18760,	2.3744212E+4,	1.3212592E+7,
0.19013,	2.3871936E+4,	1.3284937E+7,
0.19261,	2.4004296E+4,	1.3359858E+7,
0.19513,	2.4135817E+4,	1.3434247E+7,
0.19765,	2.4247269E+4,	1.3497285E+7,
0.20012,	2.4332207E+4,	1.3545267E+7,
0.20255,	2.4395464E+4,	1.3580953E+7,
0.20503,	2.4443214E+4,	1.3607826E+7,
0.20755,	2.4475122E+4,	1.3625711E+7,
0.21005,	2.4480973E+4,	1.3628774E+7,
0.21252,	2.4448240E+4,	1.3609927E+7,
0.21513,	2.4370781E+4,	1.3565733E+7,
0.21757,	2.4275725E+4,	1.3511689E+7,

TABLE 6.2.1-26 (SHEET 3 OF 7)

Time (s)	Mass Flow _(lb/s)_	Energy Flow (Btu/s)
0.22002,	2.4182117E+4,	1.3458565E+7,
0.22262,	2.4107319E+4,	1.3416180E+7,
0.22513,	2.4069692E+4,	1.3394923E+7,
0.22754,	2.4059667E+4,	1.3389325E+7,
0.23021,	2.4062445E+4,	1.3390990E+7,
0.23263,	2.4067014E+4,	1.3393646E+7,
0.23504,	2.4070423E+4,	1.3395637E+7,
0.23752,	2.4078331E+4,	1.3400188E+7,
0.24011,	2.4100562E+4,	1.3412891E+7,
0.24259,	2.4140469E+4,	1.3435579E+7,
0.24509,	2.4200933E+4,	1.3469938E+7,
0.24750,	2.4265283E+4,	1.3506417E+7,
0.25003,	2.4321768E+4,	1.3538422E+7,
0.25261,	2.4353615E+4,	1.3556377E+7,
0.25505,	2.4344223E+4,	1.3550864E+7,
0.25757,	2.4290066E+4,	1.3519957E+7,
0.26001,	2.4201115E+4,	1.3459386E+7,
0.26258,	2.4078015E+4,	1.3399541E+7,
0.26505,	2.3945320E+4,	1.3324386E+7,
0.26754,	2.3810797E+4,	1.3248271E+7,
0.27004,	2.3687905E+4,	1.3178772E+7,
0.27253,	2.3586351E+4,	1.3121419E+7,
0.27511,	2.3502079E+4,	1.3073899E+7,
0.27752,	2.3455387E+4,	1.3047656E+7,
0.28006,	2.3438152E+4,	1.3038153E+7,
0.28252,	2.3453951E+4,	1.3047366E+7,
0.28511,	2.3503649E+4,	1.3075784E+7,
0.28760,	2.3577850E+4,	1.3117988E+7,
0.29004,	2.3669172E+4,	1.3169845E+7,
0.29253,	2.3774953E+4,	1.3229813E+7,
0.29506,	2.3886183E+4,	1.3292821E+7,
0.29756,	2.3996260E+4,	1.3355146E+7,
0.30005,	2.4097381E+4,	1.3412378E+7,
0.30261,	2.4184345E+4,	1.3461533E+7,
0.30501,	2.4247804E+4,	1.3497363E+7,
0.30766,	2.4289171E+4,	1.3520597E+7,
0.31011,	2.4303098E+4,	1.3528275E+7,
0.31262,	2.4292119E+4,	1.3521774E+7,
0.31508,	2.4261540E+4,	1.3504208E+7,
0.31755,	2.4219339E+4,	1.3480093E+7,
0.32003,	2.4174469E+4,	1.3454522E+7,
0.32255,	2.4138798E+4,	1.3434171E+7,
0.32507,	2.4117832E+4,	1.3422282E+7,
0.32755,	2.4111291E+4,	1.3418580E+7,
0.33000,	2.4108026E+4,	1.3416724E+7,

TABLE 6.2.1-26 (SHEET 4 OF 7)

	Mass Flow	Energy Flow
Time (s)	<u>(lb/s)</u>	(Btu/s)
0.33255,	2.4097899E+4,	1.3410959E+7,
0.33503,	2.4079157E+4,	1.3410939E+7, 1.3400306E+7,
0.33753,	2.4059258E+4,	1.3389024E+7,
0.34000,	2.4048690E+4,	1.3383058E+7,
0.34262,	2.4055783E+4,	1.3387132E+7,
0.34511,	2.403763E+4, 2.4075206E+4,	1.3398197E+7,
0.34758,	2.4094905E+4,	1.3409386E+7,
0.35002,	2.4103796E+4,	1.3414398E+7,
0.35256,	2.4103790E+4, 2.4096385E+4,	1.3410127E+7,
0.35504,	2.4075163E+4,	1.3410127E+7, 1.3398034E+7,
	2.4045807E+4,	1.3381330E+7,
0.35762,		,
0.36008, 0.36254,	2.4017017E+4, 2.3991591E+4,	1.3364990E+7,
•	2.3972836E+4,	1.3350596E+7, 1.3340000E+7,
0.36508,	•	•
0.36762,	2.3966208E+4,	1.3336299E+7, 1.3339075E+7,
0.37000,	2.3971003E+4,	•
0.37253, 0.37510	2.3979807E+4, 2.3980967E+4,	1.3344103E+7, 1.3344761E+7,
0.37510,	•	,
0.37760, 0.38008,	2.3966909E+4,	1.3336760E+7,
•	2.3941814E+4,	1.3322495E+7,
0.38251,	2.3916742E+4,	1.3308297E+7,
0.38502,	2.3903255E+4,	1.3300691E+7,
0.38579,	2.3908591E+4,	1.3303782E+7,
0.39003,	2.3926500E+4,	1.3313992E+7,
0.39251,	2.3950989E+4,	1.3327913E+7, 1.3338405E+7,
0.39509,	2.3969493E+4,	1.3342291E+7,
0.39751, 0.40006,	2.3976395E+4, 2.3979176E+4,	1.3343952E+7,
•	•	1.3347733E+7,
0.40261, 0.40508,	2.3986081E+4, 2.3999851E+4,	1.3355527E+7,
0.40770,	2.4020182E+4,	1.3367041E+7,
0.40770, 0.41005,	•	,
,	2.4041523E+4,	1.3379124E+7,
0.41280,	2.4061453E+4,	1.3390382E+7, 1.3400786E+7,
0.41514, 0.41763,	2.4079853E+4, 2.4100740E+4,	1.3400780E+7, 1.3412580E+7,
0.41703,	2.4144685E+4,	1.3426111E+7,
•	•	•
0.42256,	2.4148532E+4,	1.3439591E+7,
0.42511,	2.4166027E+4,	1.3449429E+7,
0.42760, 0.43009,	2.4173080E+4,	1.3453348E+7,
	2.4172381E+4,	1.3452864E+7, 1.3451262E+7,
0.43256,	2.4169703E+4,	,
0.43504,	2.4168789E+4,	1.3450676E+7,
0.43778,	2.4169276E+4,	1.3450904E+7,
0.44017,	2.4165751E+4,	1.3448854E+7,
0.44263,	2.4154330E+4,	1.3442313E+7,

TABLE 6.2.1-26 (SHEET 5 OF 7)

	Mass Flow	Energy Flow
Time (s)	<u>(lb/s)</u>	(Btu/s)
0.44509,	2.4135122E+4,	1.3431355E+7,
0.44753,	2.4113758E+4,	1.3419208E+7,
0.45012,	2.4094979E+4,	1.3408512E+7,
0.45263,	2.4084151E+4,	1.3402379E+7,
0.45510,	2.4078610E+4,	1.3399245E+7,
0.45766,	2.4072112E+4,	1.3395561E+7,
0.46015,	2.4061009E+4,	1.3389283E+7,
0.46261,	2.4044888E+4,	1.3380159E+7,
0.46507,	2.4028418E+4,	1.3370837E+7,
0.46754,	2.4014861E+4,	1.3363172E+7,
0.47015,	2.4007389E+4,	1.3358971E+7,
0.47259,	2.4007662E+4,	1.3359161E+7,
0.47505,	2.4013697E+4,	1.3362618E+7,
0.47760,	2.4023721E+4,	1.3368333E+7,
0.48009,	2.4036380E+4,	1.3375569E+7,
0.48262,	2.4052248E+4,	1.3384600E+7,
0.48509,	2.4074545E+4,	1.3397290E+7,
0.48760,	2.4107246E+4,	1.3415863E+7,
0.49013,	2.4148401E+4,	1.3439192E+7,
0.49260,	2.4192806E+4,	1.3464343E+7,
0.49504,	2.4232365E+4,	1.3486726E+7,
0.49750,	2.4262844E+4,	1.3503931E+7,
0.50009,	2.4280483E+4,	1.3513829E+7,
0.51004,	2.4229105E+4,	1.3484245E+7,
0.52019,	2.4120069E+4,	1.3422231E+7,
0.53012,	2.3944743E+4,	1.3322982E+7,
0.54004,	2.3980354E+4,	1.3343661E+7,
0.55002,	2.4145342E+4,	1.3437383E+7,
0.56008,	2.4221418E+4,	1.3480274E+7,
0.57014,	2.4209329E+4,	1.3473144E+7,
0.58011,	2.4144979E+4,	1.3436505E+7,
0.59000,	2.4107243E+4,	1.3415227E+7,
0.60006,	2.4159685E+4,	1.3445105E+7,
0.61010,	2.4217996E+4,	1.3478169E+7,
0.62013,	2.4261479E+4,	1.3502731E+7,
0.63013,	2.4256432E+4,	1.3499721E+7,
0.64003,	2.4203746E+4,	1.3469774E+7,
0.65008,	2.4162745E+4,	1.3446618E+7,
0.66009,	2.4159154E+4,	1.3444706E+7,
0.67000,	2.4187283E+4,	1.3460781E+7,
0.68000,	2.4231721E+4,	1.3486025E+7,
0.69011,	2.4263450E+4,	1.3503981E+7,
0.70012,	2.4273260E+4,	1.3509453E+7,
0.71012,	2.4252390E+4,	1.3503212E+7,
0.72015,	2.4246713E+4,	1.3494349E+7,

TABLE 6.2.1-26 (SHEET 6 OF 7)

	Mass Flow	Energy Flow
Time (s)	<u>(lb/s)</u>	(Btu/s)
0.73014,	2.4255830E+4,	1.3499597E+7,
0.74019,	2.4284243E+4,	1.3515760E+7,
0.75002,	2.4295893E+4,	1.3522331E+7,
0.76012,	2.4285562E+4,	1.3516424E+7,
0.77013,	2.4270343E+4,	1.3507794E+7,
0.78001,	2.4266732E+4,	1.3505797E+7,
0.79003,	2.4279302E+4,	1.3513005E+7,
0.80011,	2.4295891E+4,	1.3522447E+7,
0.81004,	2.4310716E+4,	1.3530822E+7,
0.82001,	2.4318677E+4,	1.3535287E+7,
0.83012,	2.4318126E+4,	1.3534984E+7,
0.84019,	2.4323296E+4,	1.3537931E+7,
0.85007,	2.4333920E+4,	1.3543954E+7,
0.86001,	2.4343134E+4,	1.3549166E+7,
0.87001,	2.4345149E+4,	1.3550301E+7,
0.88004,	2.4338762E+4,	1.3546668E+7,
0.89006,	2.4335464E+4,	1.3544826E+7,
0.90005,	2.4335806E+4,	1.3545058E+7,
0.91005,	2.4340583E+4,	1.3547769E+7,
0.92020,	2.4348287E+4,	1.3552160E+7,
0.93008,	2.4355085E+4,	1.3556021E+7,
0.94000,	2.4360745E+4,	1.3559249E+7,
0.95007,	2.4367196E+4,	1.3562914E+7,
0.96009,	2.4374109E+4,	1.3566815E+7,
0.97015,	2.4378996E+4,	1.3569608E+7,
0.98005,	2.4379731E+4,	1.3570001E+7,
0.99007,	2.4378418E+4,	1.3569255E+7,
1.00010,	2.4375674E+4,	1.3567753E+7,
1.05019,	2.4380088E+4,	1.3570353E+7,
1.10005,	2.4382999E+4,	1.3572130E+7,
1.15002,	2.4383322E+4,	1.3572555E+7,
1.20009,	2.4377813E+4,	1.3569670E+7,
1.25007,	2.4374905E+4,	1.3568424E+7,
1.30007,	2.4378444E+4,	1.3570840E+7,
1.35018,	2.4377156E+4,	1.3570652E+7,
1.40010,	2.4370716E+4,	1.3567578E+7,
1.45004,	2.4382101E+4,	1.3574759E+7,
1.50024,	2.4361620E+4,	1.3563940E+7,
1.55003,	2.4349789E+4,	1.3558156E+7,
1.60012,	2.4323449E+4,	1.3544249E+7,
1.65001,	2.4297197E+4,	1.3530488E+7,
1.70009,	2.4265055E+4,	1.3513472E+7,
1.75003,	2.4230524E+4,	1.3495182E+7,
1.80007,	2.4195149E+4,	1.3476460E+7,
1.85006,	2.4154185E+4,	1.3454644E+7,

TABLE 6.2.1-26 (SHEET 7 OF 7)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
1.90005,	2.4110518E+4,	1.3431345E+7,
1.95001,	2.4061980E+4,	1.3405353E+7,
2.00002,	2.4006998E+4,	1.3375804E+7,
10.0000,	2.4006998E+4,	1.3375804E+7,

TABLE 6.2.1-26A

STEAM GENERATOR COMPARTMENT TIME DEPENDENT FLOW CONDITIONS (16 IN. FEEDWATER LINE CONDITION)

Time (s)	Mass Flow _(lb/s)_	Energy Flow _(Btu/s)_
0.0	1.3371E4	425.11
5.0	1.3371E4	425.11

TABLE 6.2.1-27 (SHEET 1 OF 36)

STEAM GENERATOR COMPARTMENT TIME DEPENDENT FLOW CONDITIONS (236-in.² BREAK)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) .00000, .00101, .00202, .00301, .00402, .00501, .00602, .00701, .00800, .01002, .01103, .01202, .01303, .01401, .01500, .01602, .01702, .01801, .02901, .02100, .02200, .02301, .02402, .02502, .02603, .02702, .02801, .02900, .03001, .03100, .03202, .03300,		
.03300, .03403, .03502, .03601, .03701, .03802, .03900,	2.5797867E+04, 2.5729553E+04, 2.5666245E+04, 2.5615037E+04, 2.5584268E+04, 2.5576045E+04, 2.5593825E+04,	1.4238755E+07, 1.4200369E+07, 1.4164838E+07, 1.4136130E+07, 1.4118888E+07, 1.4114292E+07, 1.4124256E+07,

TABLE 6.2.1-27 (SHEET 2 OF 36)

.04000, 2.5636989E+04, 1.4148441E+07, .04101, 2.5709142E+04, 1.4189612E+07, .04201, 2.5803415E+04, 1.4242315E+07, .04300, 3.9073270E+04, 2.1622832E+07, .04400, 3.6123909E+04, 1.9959244E+07, .04500, 3.7514640E+04, 2.0734863E+07, .04600, 3.7123722E+04, 2.0519158E+07, .04701, 3.7595199E+04, 2.0781432E+07, .04901, 3.8569457E+04, 2.1328762E+07, .05000, 3.9090779E+04, 2.1328762E+07, .05000, 3.9924193E+04, 2.1621351E+07, .05101, 3.9522644E+04, 2.1863598E+07, .05203, 3.9924193E+04, 2.2089344E+07, .05301, 4.0328238E+04, 2.2316935E+07, .05402, 4.0780930E+04, 2.2571622E+07, .05602, 4.1028097E+04, 2.2726524E+07, .05502, 4.1028097E+04, 2.2776867E+07, .05701, 4.0827711E+04, 2.2776867E+07, .05902, 4.0366748E+04, 2.2594210E+07, .05801, 4.0614117E+04, 2.279899E+07, .05902, 4.0306748E+04, 2.2594210E+07, .06005, 3.9705141E+04, 2.2594210E+07, .06005, 3.9705141E+04, 2.2598999E+07, .06005, 3.9705141E+04, 2.1573731E+07, .06203, 3.8043324E+04, 2.1573731E+07, .06203, 3.8043324E+04, 2.1573731E+07, .06203, 3.8043324E+04, 2.1151257E+07, .06609, 3.8043324E+04, 2.1144413E+07, .06810, 3.8431685E+04, 2.1144413E+07, .06810, 3.8681749E+04, 2.1329199E+07, .06710, 3.8681749E+04, 2.1329199E+07, .06710, 3.8681749E+04, 2.1329199E+07, .07103, 3.8694016E+04, 2.13393940E+07, .07203, 3.8694016E+04, 2.133	.04000, 2.5636989E+04, 1.4148441E+07, .04101, 2.5709142E+04, 1.4189612E+07, .04201, 2.5803415E+04, 1.4242315E+07, .04300, 3.9073270E+04, 2.1622832E+07, .04400, 3.6123909E+04, 1.9959244E+07, .04500, 3.7514640E+04, 2.0734863E+07, .04600, 3.7123722E+04, 2.0519158E+07, .04701, 3.7595199E+04, 2.0781432E+07, .04901, 3.8569457E+04, 2.1004792E+07, .05000, 3.9090779E+04, 2.1621351E+07, .05000, 3.9990779E+04, 2.1621351E+07, .05101, 3.9522644E+04, 2.1863598E+07, .05203, 3.9924193E+04, 2.2089344E+07, .05301, 4.0328238E+04, 2.2316935E+07, .05502, 4.1058245E+04, 2.2571622E+07, .05602, 4.1028097E+04, 2.2726524E+07, .05602, 4.1028097E+04, 2.2726524E+07, .05701, 4.0827711E+04, 2.257462E+07, .05902, 4.0306748E+04, 2.2473446E+07, .05902, 4.0306748E+04, 2.2473446E+07, .06005, 3.9705141E+04, 2.1958902E+07, .06101, 3.9019798E+04, 2.1151257E+07, .06101, 3.9019798E+04, 2.1151257E+07, .06203, 3.8524030E+04, 2.1151257E+07, .06306, 3.8264535E+04, 2.1151257E+07, .06306, 3.8264535E+04, 2.1151257E+07, .06402, 3.8043324E+04, 2.1027233E+07, .06506, 3.7528857E+04, 2.1151257E+07, .06609, 3.8047881E+04, 2.1151257E+07, .06609, 3.8047881E+04, 2.11347369E+07, .06710, 3.8248523E+04, 2.11347369E+07, .06904, 3.8577465E+04, 2.1247369E+07, .06710, 3.8248523E+04, 2.11347369E+07, .06710, 3.8248523E+04, 2.1337566E+07, .07103, 3.8716321E+04, 2.1387566E+07, .07103, 3.8716321E+04, 2.13406737E+07, .07103, 3.8716321E+04, 2.1406737E+07, .07103, 3.8716321E+04, 2.1406737E+07, .07103, 3.8716321E+04, 2.1406737E+07, .07103, 3.8716321E+04, 2.1406737E+07, .07103, 3.871632	Timo (s)	Mass Flow	Energy Flow
.04101,	.04101, 2.5709142E+04, 1.4189612E+07, 04201, 2.5803415E+04, 1.4242315E+07, 04300, 3.9073270E+04, 2.1622832E+07, 04400, 3.6123909E+04, 1.9959244E+07, 04500, 3.7514640E+04, 2.0734863E+07, 04701, 3.7595199E+04, 2.0734863E+07, 04701, 3.7595199E+04, 2.0781432E+07, 04801, 3.7990734E+04, 2.1004792E+07, 04901, 3.8569457E+04, 2.1328762E+07, 05000, 3.9090779E+04, 2.1621351E+07, 05101, 3.9522644E+04, 2.1863598E+07, 05203, 3.9924193E+04, 2.2089344E+07, 05301, 4.0328238E+04, 2.2316935E+07, 05402, 4.0780930E+04, 2.2571622E+07, 05502, 4.1058245E+04, 2.276524E+07, 05502, 4.1058245E+04, 2.277652E+07, 05502, 4.1058245E+04, 2.2770867E+07, 05701, 4.0827711E+04, 2.2594210E+07, 05801, 4.0614117E+04, 2.2594210E+07, 06905, 3.9705141E+04, 2.2298999E+07, 06101, 3.9019798E+04, 2.1573731E+07, 06203, 3.8524030E+04, 2.1596608E+07, 06306, 3.8264535E+04, 2.1151257E+07, 06306, 3.8043324E+04, 2.1926608E+07, 06506, 3.7528857E+04, 2.1926608E+07, 06609, 3.8043324E+04, 2.1927338E+07, 06610, 3.8248523E+04, 2.1151257E+07, 06506, 3.7528857E+04, 2.1926608E+07, 06710, 3.8641749E+04, 2.193996E+07, 06710, 3.8248523E+04, 2.1151257E+07, 06506, 3.7528857E+04, 2.1151257E+07, 06506, 3.8043324E+04, 2.1151257E+07, 06506, 3.854358E+04, 2.1151257E+07, 06506, 3.854358E+04, 2.1151257E+07, 06506, 3.854358E+04, 2.1151257E+07, 06506, 3.854535E+04, 2.1151257E+07, 06506, 3.854553E+04, 2.1151257E+07, 06506, 3.854535E+04, 2.1151257E+07, 06506, 3.854553E+04, 2.1154558E+07, 07508, 3.864553E+04, 2.1154558E+07, 07508, 3.864553E+04, 2.1154558E+07,	<u>111116 (3)</u>	<u> (ID/S)</u>	(Dtu/5)
.07300, 3.0010314E+04, 2.1340303E+07,	.07508,3.8286475E+04,2.1164259E+07,.07601,3.8094753E+04,2.1056554E+07,.07705,3.7869887E+04,2.0930259E+07,.07808,3.7625546E+04,2.0793040E+07,.07903,3.7377536E+04,2.0653886E+07,	.04101, .04201, .04300, .04400, .04500, .04600, .04701, .04801, .05000, .05101, .05203, .05301, .05402, .05502, .05602, .05701, .05801, .06904, .06101, .06203, .06306, .06402, .06506, .06609, .06710, .06810, .06904, .07010, .07103, .07203, .07308,	2.5636989E+04, 2.5709142E+04, 2.5803415E+04, 3.9073270E+04, 3.6123909E+04, 3.7514640E+04, 3.7515199E+04, 3.7990734E+04, 3.9990779E+04, 3.9990779E+04, 3.9924193E+04, 4.0780930E+04, 4.1058245E+04, 4.1058245E+04, 4.0827711E+04, 4.0614117E+04, 4.0306748E+04, 3.9705141E+04, 3.9705141E+04, 3.9019798E+04, 3.8524030E+04, 3.8524030E+04, 3.8524030E+04, 3.8524030E+04, 3.8524030E+04, 3.8524030E+04, 3.8524030E+04, 3.8524030E+04, 3.8524030E+04, 3.854535E+04, 3.8047881E+04, 3.8047881E+04, 3.8431685E+04, 3.8431685E+04, 3.8577465E+04, 3.8681749E+04, 3.8694016E+04, 3.8694016E+04, 3.8694016E+04,	(Btu/s) 1.4148441E+07, 1.4189612E+07, 1.4242315E+07, 2.1622832E+07, 1.9959244E+07, 2.0734863E+07, 2.0519158E+07, 2.0781432E+07, 2.1004792E+07, 2.1328762E+07, 2.1621351E+07, 2.1863598E+07, 2.289344E+07, 2.2316935E+07, 2.2571622E+07, 2.2726524E+07, 2.2726524E+07, 2.2707867E+07, 2.2707867E+07, 2.298999E+07, 2.1958902E+07, 2.1958902E+07, 2.1151257E+07, 2.1958902E+07, 2.1151257E+07, 2.1027233E+07, 2.1027233E+07, 2.1031529E+07, 2.1144413E+07, 2.1349199E+07, 2.1387566E+07, 2.1393940E+07, 2.1393940E+07, 2.1393940E+07, 2.1393940E+07, 2.1393940E+07, 2.1393940E+07, 2.1346583E+07,
.U/JUO. J.OU UJ 4ETU4. Z. 1340303ETU/.	.07404, 3.8477661E+04, 2.1271803E+07, .07508, 3.8286475E+04, 2.1164259E+07, .07601, 3.8094753E+04, 2.1056554E+07, .07705, 3.7869887E+04, 2.0930259E+07, .07808, 3.7625546E+04, 2.0793040E+07, .07903, 3.7377536E+04, 2.0653886E+07,	.07103, .07203,	3.8716321E+04, 3.8694016E+04,	2.1406737E+07, 2.1393940E+07,

TABLE 6.2.1-27 (SHEET 3 OF 36)

TABLE 6.2.1-27 (SHEET 4 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
.23004, .24009, .25004, .26004, .27000, .28002, .29009, .30013, .31010, .32015, .33008, .34008, .35014, .36003, .37004, .38005, .39001, .40001, .41008, .42013, .43005, .44011, .45002, .46007, .47014, .48003, .49004, .50014, .51004, .52016, .53013, .54020, .55003,	3.5231527E+04, 3.5109052E+04, 3.5068150E+04, 3.5121658E+04, 3.5159026E+04, 3.5246849E+04, 3.5606682E+04, 3.5610831E+04, 3.5731357E+04, 3.5406465E+04, 3.5406465E+04, 3.5428013E+04, 3.5316627E+04, 3.5402063E+04, 3.5506771E+04, 3.5587259E+04, 3.5587259E+04, 3.5458889E+04, 3.5458889E+04, 3.5389368E+04, 3.5331713E+04, 3.5344742E+04, 3.5344742E+04, 3.5455419E+04, 3.5456311E+04, 3.4956311E+04, 3.4967429E+04,	(Btu/s) 1.9454787E+07, 1.9386133E+07, 1.9363427E+07, 1.9393549E+07, 1.9414539E+07, 1.9463869E+07, 1.9665651E+07, 1.9947367E+07, 1.9819724E+07, 1.9552039E+07, 1.9550425E+07, 1.9550425E+07, 1.9550441E+07, 1.9550400E+07, 1.9669110E+07, 1.9677021E+07, 1.9687547E+07, 1.9687547E+07, 1.9687547E+07, 1.9653881E+07, 1.9582016E+07, 1.9582016E+07, 1.9543272E+07, 1.9543272E+07, 1.9511278E+07, 1.9511278E+07, 1.9580555E+07, 1.9580555E+07, 1.9580555E+07, 1.9580555E+07, 1.9301100E+07, 1.9307806E+07, 1.9307806E+07, 1.9307806E+07, 1.9423487E+07,
.56006, .57007, .58008, .59009, .60004, .61004, .62003,	3.5249947E+04, 3.5274526E+04, 3.5192533E+04, 3.5173286E+04, 3.5207910E+04, 3.5263128E+04, 3.5289369E+04,	1.9466290E+07, 1.9480020E+07, 1.9434100E+07, 1.9423484E+07, 1.9443066E+07, 1.9474124E+07, 1.9488901E+07,
.63008,	3.5280693E+04,	1.9484094E+07,

TABLE 6.2.1-27 (SHEET 5 OF 36)

Time (a)	Mass Flow	Energy Flow
<u> 1 ii iie (S)</u>	<u>(ID/S)</u>	<u>(Blu/S)</u>
Time (s) .64011, .65018, .66010, .67011, .68006, .69010, .70027, .71002, .72011, .73008, .74015, .75007, .76011, .77010, .78003, .80009, .81007, .82009, .83007, .84002, .85010, .86007, .87000, .88005, .89000, .90011, .91008, .92015, .93004, .94007	(lb/s) 3.5205381E+04, 3.5185255E+04, 3.5247094E+04, 3.5294402E+04, 3.5270057E+04, 3.5224067E+04, 3.5205346E+04, 3.5205346E+04, 3.5204349E+04, 3.5216969E+04, 3.5278490E+04, 3.5294514E+04, 3.53294514E+04, 3.5324933E+04, 3.5349418E+04, 3.5337390E+04, 3.5315749E+04, 3.5315673E+04, 3.5398275E+04, 3.5471641E+04, 3.5544815E+04, 3.5574072E+04, 3.5609336E+04, 3.5609336E+04,	(Btu/s) 1.9441981E+07, 1.9430942E+07, 1.9465823E+07, 1.9492475E+07, 1.9478927E+07, 1.9453332E+07, 1.9443190E+07, 1.9443150E+07, 1.9443007E+07, 1.9450347E+07, 1.9468535E+07, 1.9485325E+07, 1.9511865E+07, 1.9511865E+07, 1.9511366E+07, 1.9520270E+07, 1.9520270E+07, 1.9509200E+07, 1.9509200E+07, 1.9509200E+07, 1.9509200E+07, 1.9569725E+07, 1.9597672E+07, 1.9639309E+07, 1.9639309E+07, 1.9639309E+07, 1.9676169E+07, 1.9676169E+07, 1.9696218E+07,
.89000, .90011, .91008, .92015, .93004, .94007, .95006, .96004,	3.5514959E+04, 3.5544815E+04, 3.5574072E+04, 3.5609336E+04, 3.5644462E+04, 3.5681054E+04, 3.5708010E+04, 3.5732460E+04,	1.9622268E+07, 1.9639309E+07, 1.9656051E+07, 1.9676169E+07, 1.9696218E+07, 1.9717092E+07, 1.9732566E+07, 1.9746650E+07,
.97008, .98003, .99019, 1.00012, 1.05008, 1.10007, 1.15002, 1.20007, 1.25001,	3.5767399E+04, 3.5808071E+04, 3.5852729E+04, 3.5895623E+04, 3.6112647E+04, 3.6338443E+04, 3.6608642E+04, 3.6888439E+04, 3.7127241E+04,	1.9766649E+07, 1.9789841E+07, 1.9815310E+07, 1.9839774E+07, 1.9963381E+07, 2.0092128E+07, 2.0245910E+07, 2.0405162E+07, 2.0541579E+07,

TABLE 6.2.1-27 (SHEET 6 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) 1.30009, 1.35002, 1.40001, 1.45005, 1.50008, 1.55002, 1.60009, 1.65015, 1.70001, 1.73004, 1.80009, 1.85004, 1.90007, 1.95004, 2.05008, 2.15010, 2.25002, 2.30000, 2.35011, 2.40001, 2.45002, 2.50011, 2.45002, 2.50011, 2.55008, 2.60014, 2.65001, 2.75018, 2.80003, 2.85001, 2.90018,		(Btu/s) 2.0656265E+07, 2.0747071E+07, 2.0812441E+07, 2.0870059E+07, 2.0910243E+07, 2.1009055E+07, 2.1045152E+07, 2.1148978E+07, 2.1196588E+07, 2.1237084E+07, 2.1237084E+07, 2.1305277E+07, 2.1308836E+07, 2.1302004E+07, 2.1302004E+07, 2.130927E+07, 2.1175694E+07, 2.1175694E+07, 2.1024926E+07, 2.0929322E+07, 2.0796234E+07, 2.0677130E+07, 2.0570251E+07, 2.0497962E+07, 2.0499962E+07, 2.0499962E+07
2.95009, 3.00005,	3.4981705E+04, 3.4718072E+04,	1.9432201E+07, 1.9289429E+07,

TABLE 6.2.1-27 (SHEET 7 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
.00000, .00101, .00201, .00301, .00401, .00500, .00602, .00700, .00802, .00903, .01003, .01101, .01203, .01502, .01605, .01703, .01502, .01605, .01703, .01901, .02004, .02103, .02204, .02403, .02502, .02607, .02707, .02807, .02906, .03005, .03107, .03207, .03207, .03305,	1.0904418E+04, 2.6783296E+04, 2.6059905E+04, 2.6630723E+04, 2.6299962E+04, 2.6349622E+04, 2.6349622E+04, 2.6380583E+04, 2.6380583E+04, 2.6380531E+04, 2.6428212E+04, 2.6455432E+04, 2.6494736E+04, 2.6507040E+04, 2.6614296E+04, 2.6873875E+04, 2.6873875E+04, 2.7467003E+04, 2.9703593E+04, 3.0491958E+04, 3.0380016E+04, 3.0268393E+04, 2.9902195E+04, 2.99394309E+04, 2.9276976E+04, 2.9276976E+04, 2.9210971E+04, 2.9233801E+04, 2.9233801E+04, 2.9300991E+04, 2.9433417E+04, 2.9433417E+04, 2.9433417E+04,	(Btu/s) 6.9546146E+06, 1.7817432E+07, 1.6564051E+07, 1.6923731E+07, 1.6714564E+07, 1.6797431E+07, 1.6749019E+07, 1.6768774E+07, 1.6755933E+07, 1.6765424E+07, 1.6771220E+07, 1.6813421E+07, 1.6830145E+07, 1.6830535E+07, 1.6846432E+07, 1.6846432E+07, 1.68930145E+07, 1.7032259E+07, 1.7463110E+07, 1.7463110E+07, 1.89401975E+07, 1.9461919E+07, 1.9461919E+07, 1.9461919E+07, 1.9461919E+07, 1.9461919E+07, 1.9581326E+07, 1.8702756E+07, 1.8702756E+07, 1.8590523E+07, 1.8590523E+07, 1.8689301E+07, 1.8719351E+07, 1.8719351E+07, 1.8719351E+07, 1.8818687E+07,
.03400, .03507, .03603, .03700, .03803, .03906, .04000,	2.9757005E+04, 2.9977287E+04, 3.0194624E+04, 3.0444230E+04, 3.0748400E+04, 3.1197470E+04, 3.1497312E+04,	1.8927234E+07, 1.9068795E+07, 1.9208524E+07, 1.9369022E+07, 1.9364717E+07, 1.9795836E+07, 2.0047030E+07,

TABLE 6.2.1-27 (SHEET 8 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) .04100, .04205, .04305, .04406, .04507, .04602, .04700, .04808, .04909, .05001, .05109, .05206, .05300, .05401, .05501, .05611, .05711, .05811, .05914, .06007, .06130, .06200, .06308, .06401, .06504, .06605, .06700, .06803, .06903, .07008, .07106, .07201, .07206, .07409, .07507, .07607, .07607, .07607, .07607, .07607, .07715, .07814,		
.07900, .08009, .08107,	3.1157700E+04, 3.1183245E+04, 3.1200280E+04,	1.9823092E+07, 1.9839430E+07, 1.9850340E+07,

TABLE 6.2.1-27 (SHEET 9 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
.08206, .08308, .08404, .08501, .08601, .08709, .08804, .08901, .09089, .09110, .09201, .09301, .09409, .09501, .09600, .09713, .09814, .09912, .10516, .11505, .12505, .13504, .14509, .15504, .16002, .16506, .17501, .18001,	3.1203954E+04, 3.1193945E+04, 3.1170566E+04, 3.1134922E+04, 3.1087116E+04, 3.1029503E+04, 3.0974094E+04, 3.0974094E+04, 3.0853531E+04, 3.0743409E+04, 3.0685573E+04, 3.0573333E+04, 3.0573333E+04, 3.0573333E+04, 3.0516623E+04, 3.0452875E+04, 3.0300946E+04, 3.0300946E+04, 2.9947775E+04, 2.9827896E+04, 2.9681357E+04, 2.9827896E+04, 2.9681357E+04, 2.9125707E+04, 2.8981100E+04, 2.898110E+04, 2.89811E+04, 2.89811E+04, 2.89811E+04, 2.89811E+04,	(Btu/s) 1.9852600E+07, 1.9846042E+07, 1.9838001E+07, 1.9807821E+07, 1.9776942E+07, 1.9739000E+07, 1.9704012E+07, 1.9667644E+07, 1.9626200E+07, 1.9555347E+07, 1.9518020E+07, 1.9478562E+07, 1.9478562E+07, 1.9489168E+07, 1.9334105E+07, 1.9334105E+07, 1.9301972E+07, 1.9301972E+07, 1.9301972E+07, 1.9301972E+07, 1.9301972E+07, 1.8967496E+07, 1.8967496E+07, 1.8967496E+07, 1.8873476E+07, 1.8873476E+07, 1.88178459E+07, 1.8217077E+07, 1.8178459E+07, 1.8178459E+07, 1.7041673E+07, 1.7041673E+07, 1.7647852E+07, 1.7647852E+07, 1.7429901E+07, 1.7283536E+07,
.18504, .19002, .19500, .20018, .21000, .22013,	2.6715030E+04, 2.6493250E+04, 2.6124672E+04, 2.5841963E+04, 2.5138580E+04, 2.4375517E+04,	1.6977152E+07, 1.6778063E+07, 1.6682797E+07, 1.6419880E+07, 1.5970348E+07, 1.5482003E+07,

TABLE 6.2.1-27 (SHEET 10 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
.23016, .24014, .25007, .26013, .27011, .28001, .29006, .30001, .31009, .32009, .33014, .34090, .35014, .36013, .37011, .38014, .39882, .40019, .41010, .42003, .43008, .44008, .45083, .46002, .47010, .48024, .49003, .50003, .50003, .51002,	2.3823925E+04, 2.3369722E+04, 2.2929182E+04, 2.2691429E+04, 2.2497616E+04, 2.2356313E+04, 2.2180437E+04, 2.2180437E+04, 2.1944465E+04, 2.1706070E+04, 2.1612880E+04, 2.1490487E+04, 2.1490487E+04, 2.112816E+04, 2.112816E+04, 2.1112816E+04, 2.0924470E+04, 2.0924470E+04, 2.0889126E+04, 2.0723134E+04, 2.072314E+04, 2.072	(Btu/s) 1.5130751E+07, 1.4049362E+07, 1.4559762E+07, 1.4408545E+07, 1.4205460E+07, 1.4196081E+07, 1.4004375E+07, 1.4016719E+07, 1.3935290E+07, 1.3734560E+07, 1.3725889E+07, 1.3648525E+07, 1.3586161E+07, 1.3583657E+07, 1.3583657E+07, 1.3454095E+07, 1.3494095E+07, 1.3294601E+07, 1.3294601E+07, 1.3213971E+07, 1.3113971E+07, 1.3113971E+07, 1.3920890E+07, 1.3920890E+07, 1.2995489E+07, 1.2995489E+07, 1.2994073E+07, 1.2940015E+07, 1.2940015E+07, 1.2940015E+07, 1.2924773E+07,
.52010, .53003, .54001,	2.0267614E+04, 2.0235873E+04, 2.0205732E+04,	1.2902050E+07, 1.2885929E+07, 1.2869978E+07,
.55014, .56001, .57001, .58013, .59003, .60003, .61809, .62004,	2.0176703E+04, 2.0151311E+04, 2.0126861E+04, 2.0095633E+04, 2.0062537E+04, 2.0030307E+04, 2.0002602E+04, 1.9977344E+04, 1.9955914E+04,	1.2854603E+07, 1.2841861E+07, 1.2020991E+07, 1.2012343E+07, 1.2794139E+07, 1.2776336E+07, 1.2761137E+07, 1.2747300E+07, 1.2735706E+07,

TABLE 6.2.1-27 (SHEET 11 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) .64002, .65013, .66007, .67006, .68007, .69009, .70009, .71012, .72004, .73002, .74012, .75003, .76010, .77003, .78023, .79001, .80001, .81010, .82005, .83011, .84102, .85005, .86809, .87017, .88007, .89004, .90001, .91004, .92005, .93003, .94097, .95039, .96003, .97010, .98004,	1.9937743E+04, 1.9919953E+04, 1.9892660E+04, 1.9861153E+04, 1.9846173E+04, 1.9830751E+04, 1.9816368E+04, 1.9805505E+04, 1.9795400E+04, 1.9795400E+04, 1.9772434E+04, 1.9762966E+04, 1.9759570E+04, 1.97553590E+04, 1.9755388E+04, 1.9755388E+04, 1.9756346E+04, 1.9756346E+04, 1.9756346E+04, 1.97563788E+04, 1.9763788E+04, 1.9763788E+04, 1.9772159E+04, 1.9780392E+04, 1.9780558E+04, 1.9780558E+04, 1.9780558E+04, 1.9780558E+04, 1.9780558E+04, 1.9837670E+04, 1.9837670E+04, 1.9837670E+04, 1.9828011E+04, 1.9828011E+04, 1.9841004E+04, 1.9841004E+04, 1.9869443E+04,	
.98004, .99014, 1.00014, 1.05808, 1.10006, 1.15083,	1.9899603E+04, 1.9913539E+04, 1.9927001E+04, 2.0003035E+04, 2.0001506E+04, 2.0159717E+04,	1.2607300E+07, 1.2611643E+07, 1.2616035E+07, 1.2648624E+07, 1.2666790E+07, 1.2683963E+07,

TABLE 6.2.1-27 (SHEET 12 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) 1.23001, 1.25006, 1.30002, 1.35006, 1.40006, 1.45001, 1.55006, 1.60003, 1.65008, 1.70001, 1.75004, 1.83007, 1.85009, 1.90010, 1.95302, 2.00011, 2.05007, 2.10001, 2.15006, 2.20014, 2.25007, 2.30001, 2.35006, 2.40002, 2.45090,	(lb/s) 2.0229722E+04, 2.0271963E+04, 2.0329809E+04, 2.0445870E+04, 2.0595390E+04, 2.0797697E+04, 2.2597603E+04, 2.1727914E+04, 2.2569311E+04, 2.2569311E+04, 2.7069624E+04, 3.1111373E+04, 3.8141272E+04, 3.8279306E+04, 3.8503591E+04, 3.8724291E+04, 3.1050715E+04, 3.1246237E+04, 3.1427902E+04, 3.1427902E+04, 3.1427902E+04, 3.1427902E+04, 3.1406946E+04, 3.1352832E+04, 3.1352832E+04, 3.1247389E+04, 3.1046631E+04,	(Btu/s) 1.2703628E+07, 1.2714947E+07, 1.2733996E+07, 1.2730469E+07, 1.2973763E+07, 1.4130098E+07, 1.3531673E+07, 1.3603610E+07, 1.3603610E+07, 1.6559945E+07, 1.8156965E+07, 1.8901704E+07, 1.8262443E+07, 1.8347891E+07, 1.8593102E+07, 1.8793001E+07, 1.8937338E+07, 1.9017557E+07, 1.9070014E+07, 1.8997013E+07, 1.8997013E+07, 1.8997013E+07, 1.8997013E+07, 1.8979205E+07, 1.8979205E+07, 1.8921466E+07, 1.8900368E+07,
2.40002,	3.1247389E+04,	1.8921466E+07,
2.90004, 2.95002, 3.00010,	2.7149613E+04, 2.6945061E+04, 2.6740150E+04,	1.6473829E+07, 1.6376299E+07, 1.6273687E+07,

TABLE 6.2.1-27 (SHEET 13 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) .00000, .00102, .00200, .00300, .00401, .00501, .00601, .00703, .00801, .00901, .01102, .01102, .01102, .01503, .01606, .01703, .01801, .01900, .02004, .02101, .02203, .02386, .02406, .02503, .02602, .02705, .02805, .02901, .03007, .03101, .03200, .03302, .03408, .03507, .03602,		
.03707, .03805,	3.3116108E+04, 3.3150502E+04,	1.8251151E+07, 1.8270439E+07,

TABLE 6.2.1-27 (SHEET 14 OF 36)

	Mass Flow	Energy Flow
Time (s)	<u>(lb/s)</u>	(Btu/s)
00004	0.04070005 : 04	4 00005005 : 07
.03901,	3.3197302E+04,	1.8296529E+07,
.04005,	3.3242461E+04,	1.8321653E+07,
.04103,	3.3275084E+04,	1.8340223E+07,
.04206,	3.3299100E+04,	1.8353138E+07,
.04304,	3.3307218E+04,	1.8357510E+07,
.04401,	3.3505517E+04,	1.8470440E+07,
.04502,	3.4449731E+04,	1.8992921E+07,
.04601,	3.5262496E+04,	1.9440836E+07,
.04702,	3.5879893E+04,	1.9779923E+07,
.04806,	3.6236755E+04,	1.9972897E+07,
.04900,	3.6269583E+04,	1.9985931E+07,
.05004,	3.6060590E+04,	1.9867704E+07,
.05102,	3.5924401E+04,	1.9795340E+07,
.05201, .05306,	3.6126195E+04,	1.9912190E+07, 2.0161982E+07,
.05406,	3.6575781E+04, 3.6849965E+04,	2.0310129E+07,
.05506,	3.6897666E+04,	2.0333794E+07,
.05602,	3.6882889E+04,	2.0331701E+07,
.05705,	3.6999489E+04,	2.0392556E+07,
.05805,	3.7219065E+04,	2.0515462E+07,
.05900,	3.7476960E+04,	2.0658312E+07,
.06006,	3.7707201E+04,	2.0794494E+07,
.06103,	3.7832928E+04,	2.0852950E+07,
.06203,	3.7925042E+04,	2.0904186E+07,
.06364,	3.8068676E+04,	2.0985074E+07,
.06406,	3.8264791E+04,	2.1094051E+07,
.06503,	3.8403280E+04,	2.1169090E+07,
.06604,	3.8394878E+04,	2.1161822E+07,
.06701,	3.8260995E+04,	2.1085868E+07,
.06800,	3.8071294E+04,	2.0980414E+07,
.06906,	3.7910358E+04,	2.0892004E+07,
.07002,	3.7831074E+04,	2.0848952E+07,
.07103,	3.7803983E+04,	2.0834629E+07,
.07206,	3.7811991E+04,	2.0839661E+07,
.07308,	3.7856688E+04,	2.0865381E+07,
.07402,	3.7957543E+04,	2.0922723E+07,
.07507,	3.8165873E+04,	2.1040090E+07,
.07608,	3.8451658E+04,	2.1200016E+07,
.07702,	3.8764481E+04,	2.1374287E+07,
.07808,	3.9132485E+04,	2.1579074E+07,

TABLE 6.2.1-27 (SHEET 15 OF 36)

	Mass Flow	Energy Flow
Time (s)	<u>(lb/s)</u>	(Btu/s)
.=	0.0400000= 0.4	0.4==0.40.4= 0=
.07906,	3.9492063E+04,	2.1779484E+07,
.08001,	3.9879446E+04,	2.1995774E+07,
.08103,	4.0353849E+04,	2.2260672E+07,
.08211,	4.0908980E+04,	2.2570087E+07,
.08308,	4.1412318E+04,	2.2850120E+07,
.26011, .27014,	4.0316515E+04,	2.2239430E+07, 2.2097363E+07,
.27014, .28016,	4.0058097E+04, 3.9818692E+04,	2.2097303E+07, 2.1967568E+07,
.29008,	3.9879564E+04,	2.2005338E+07,
.30006,	4.0093220E+04,	2.2127686E+07,
.31003,	4.0207611E+04,	2.2194375E+07,
.32005,	4.0201445E+04,	2.2194242E+07,
.33030,	4.0216076E+04,	2.2206028E+07,
.34000,	4.0219714E+04,	2.2211666E+07,
.35014,	4.0145695E+04,	2.2174008E+07,
.36008,	3.9994375E+04,	2.2093549E+07,
.37012,	3.9893347E+04,	2.2041597E+07,
.38021,	3.9907754E+04,	2.2054342E+07,
.39002,	3.9952948E+04,	2.2083866E+07,
.40002,	3.9935496E+04,	2.2078383E+07,
.41003,	3.9923464E+04,	2.2076291E+07,
.42007,	3.9978870E+04,	2.2111915E+07,
.43000,	4.0047123E+04,	2.2154536E+07,
.44009,	4.0054216E+04,	2.2163031E+07,
.45011,	4.0115169E+04,	2.2202630E+07,
.46005,	4.0211520E+04,	2.2260985E+07,
.47009, .48008,	4.0184188E+04, 4.0117447E+04,	2.2250703E+07, 2.2218402E+07,
.49011,	4.0013181E+04,	2.2216402E+07, 2.2165700E+07,
.50003,	4.0008453E+04,	2.2168665E+07,
.51004,	4.0050068E+04,	2.2197559E+07,
.52010,	4.0110299E+04,	2.2236690E+07,
.53007,	4.0097367E+04,	2.2234944E+07,
.54009,	4.0068467E+04,	2.2224382E+07,
.55008,	4.0018860E+04,	2.2202150E+07,
.56003,	3.9933661E+04,	2.2160059E+07,
.57006,	3.9836338E+04,	2.2111598E+07,
.58006,	3.9806384E+04,	2.2100931E+07,
.59003,	3.9786229E+04,	2.2095666E+07,
.60003,	3.9758272E+04,	2.2086180E+07,

TABLE 6.2.1-27 (SHEET 16 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
.61011, .62019, .63005, .64013, .65008, .66007, .67001, .68024, .69004, .70003, .71001, .72008, .73012, .74000, .75012, .76006, .77002, .78001, .79014, .80009, .81009, .82009, .83007, .84007, .85001, .85001, .86003, .87009, .88011, .89010, .90001, .91006, .92001, .93010, .94008,	3.9733403E+04, 3.9687180E+04, 3.9633316E+04, 3.9595001E+04, 3.95944578E+04, 3.9415717E+04, 3.9345246E+04, 3.9272848E+04, 3.9101670E+04, 3.9101670E+04, 3.898046E+04, 3.898046E+04, 3.898046E+04, 3.8980540E+04, 3.8683025E+04, 3.8595052E+04, 3.8595052E+04, 3.8595052E+04, 3.8379224E+04, 3.8379224E+04, 3.8379224E+04, 3.8379224E+04, 3.8302612E+04, 3.8302612E+04, 3.8302612E+04, 3.83030601E+04, 3.87979574E+04, 3.7979574E+04, 3.77917809E+04, 3.7789108E+04, 3.7734890E+04,	(Btu/s) 2.2078365E+07, 2.2058478E+07, 2.2034454E+07, 2.2019153E+07, 2.1996974E+07, 2.1966303E+07, 2.1937149E+07, 2.1869592E+07, 2.1869592E+07, 2.1860313E+07, 2.1792552E+07, 2.1769073E+07, 2.1741096E+07, 2.1741096E+07, 2.175378E+07, 2.1636330E+07, 2.1550884E+07, 2.1550884E+07, 2.15507415E+07, 2.1447517E+07, 2.1446716E+07, 2.1447517E+07, 2.14487E+07, 2.1442006E+07, 2.1313324E+07, 2.1313324E+07, 2.1291442E+07, 2.1291442E+07, 2.1206416E+07, 2.1238829E+07, 2.1206416E+07, 2.1176988E+07, 2.1176988E+07, 2.1151639E+07,
.95003, .96004, .97008, .98008, .99001,	3.7684520E+04, 3.7635988E+04, 3.7584676E+04, 3.7531491E+04, 3.7479837E+04,	2.1128412E+07, 2.1106175E+07, 2.1082300E+07, 2.1057291E+07, 2.1033066E+07,

TABLE 6.2.1-27 (SHEET 17 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) 1.00009, 1.05001, 1.10010, 1.15010, 1.20002, 1.25003, 1.30009, 1.35004, 1.40001, 1.45005, 1.50008, 1.55004, 1.60008, 1.65008, 1.70019, 1.75005, 1.80008, 1.85010, 1.90003, 1.95006, 2.00088, 2.05010, 2.10010, 2.15027, 2.20002, 2.25003, 2.30003, 2.35003, 2.40012, 2.45007, 2.50003, 2.55003, 2.60006, 2.65008, 2.70007, 2.75001, 2.80004,		
2.85006, 2.90035, 2.95008, 3.00006,	3.3531241E+04, 3.2924740E+04, 3.2356095E+04, 3.1975899E+04,	1.8903699E+07, 1.8661662E+07, 1.8353048E+07, 1.8153529E+07,

TABLE 6.2.1-27 (SHEET 18 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) .00000, .00102, .00201, .00302, .00400, .00503, .00603, .00703, .00804, .00902, .01003, .01102, .01200, .01302, .01400, .01500, .01600, .01701, .01802, .01905, .02005, .02103, .02205, .02103, .02205, .02103, .02205, .02301, .02402, .02504, .02601, .02706, .02806, .02902, .03001, .03103, .03205, .03302, .03407,		
.03407, .03503, .03601, .03708, .03803, .03902,	3.3738721E+04, 3.3867245E+04, 3.3840843E+04, 3.3661255E+04, 3.3438771E+04, 3.3247515E+04,	1.8599652E+07, 1.8670587E+07, 1.8654958E+07, 1.8554098E+07, 1.8429893E+07, 1.8323650E+07,

TABLE 6.2.1-27 (SHEET 19 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) .04001, .04102, .04204, .04300, .04403, .04505, .04602, .04705, .04804, .05005, .05106, .05203, .05305, .05408, .05503, .05604, .05702, .05801, .05903, .06005, .06105, .06205, .06300, .06402, .06508, .06608, .06608, .06702, .06808, .06905, .07000, .07103, .07202, .07308, .07404,		
.07404, .07506, .07607, .07705, .07800, .07905,	3.9061995E+04, 3.9923806E+04, 4.0654471E+04, 4.1238952E+04, 4.1713347E+04,	2.1546568E+07, 2.2025178E+07, 2.2430038E+07, 2.2753206E+07, 2.3014682E+07,

TABLE 6.2.1-27 (SHEET 20 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
.08005, .08105, .08210, .08310, .08404, .08501, .08611, .08704, .08807, .08909, .09009, .09110, .09211, .09310, .09411, .09511, .09603, .09706, .09803, .09706, .10504, .11501, .11501, .11501, .12002, .12508, .13011, .13509, .14009, .14508, .15011, .15500, .16006,	4.2016658E+04, 4.2198966E+04, 4.2279767E+04, 4.2281321E+04, 4.2239756E+04, 4.2177672E+04, 4.2104329E+04, 4.2052709E+04, 4.202811E+04, 4.2025770E+04, 4.2124772E+04, 4.2172432E+04, 4.2175533E+04, 4.2175533E+04, 4.1961821E+04, 4.178332E+04, 4.1339159E+04, 4.1339159E+04, 4.1143922E+04, 4.1030160E+04, 4.1277515E+04, 4.1684568E+04, 4.0861846E+04, 4.010925E+04, 4.0672513E+04, 4.0672513E+04, 4.0672513E+04, 4.0129215E+04, 4.0128710E+04, 4.0128710E+04, 4.0036140E+04, 3.9694514E+04, 3.9694514E+04,	(Btu/s) 2.3181200E+07, 2.3280508E+07, 2.3323496E+07, 2.3322819E+07, 2.3298599E+07, 2.3263276E+07, 2.3193003E+07, 2.3175363E+07, 2.3178449E+07, 2.3200843E+07, 2.3259759E+07, 2.3260295E+07, 2.3219849E+07, 2.3219849E+07, 2.3219849E+07, 2.3219849E+07, 2.2619962E+07, 2.2790489E+07, 2.2760967E+07, 2.2760967E+07, 2.2522864E+07, 2.2522864E+07, 2.2522864E+07, 2.2107457E+07, 2.2200062E+07, 2.2423415E+07, 2.2304174E+07, 2.2119836E+07, 2.2120539E+07, 2.21068352E+07, 2.1878076E+07, 2.1878076E+07, 2.1755828E+07,
.16006, .16505, .17002, .17509, .18011, .18516, .19010, .19500,	3.9472130E+04, 3.9703270E+04, 3.9968584E+04, 3.9622431E+04, 3.9104870E+04, 3.9013925E+04, 3.9202980E+04, 3.9485145E+04,	2.1755828E+07, 2.1886290E+07, 2.2033203E+07, 2.1838486E+07, 2.1550930E+07, 2.1502278E+07, 2.1608665E+07, 2.1766258E+07,
. 19000,	3.3403143⊏™04,	2.1700230⊑₹07,

TABLE 6.2.1-27 (SHEET 21 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
.20002, .21006, .22010, .23003, .24000, .25003, .26002, .27003, .28007, .29010, .30007, .31006, .32012, .33004, .34005, .35017, .36001, .37009, .38001,	(lb/s) 3.9794566E+04, 4.0042339E+04, 4.0034978E+04, 4.0069032E+04, 3.9921154E+04, 3.9753266E+04, 3.9661005E+04, 3.9609295E+04, 3.9784416E+04, 3.9855442E+04, 3.9954873E+04, 3.99551025E+04, 3.9952792E+04, 3.9962792E+04, 3.9982825E+04, 3.9882825E+04, 3.9819271E+04, 3.9729809E+04,	(Btu/s) 2.1938705E+07, 2.2075738E+07, 2.2071672E+07, 2.2090788E+07, 2.2008323E+07, 2.1915694E+07, 2.1865181E+07, 2.1837809E+07, 2.1936634E+07, 2.1977128E+07, 2.2001107E+07, 2.2035214E+07, 2.2053889E+07, 2.2036286E+07, 2.2036286E+07, 2.2044922E+07, 2.2030225E+07, 2.2004858E+07, 2.1971978E+07, 2.1931179E+07, 2.1928009E+07,
.40014, .41008, .42008, .43006, .44011, .45004, .46002, .47001, .48010, .49004, .50003, .51013, .52000, .53008, .54009, .55003, .56000, .57007, .58002,	3.9790731E+04, 3.9848676E+04, 3.9901479E+04, 3.9943806E+04, 3.9890825E+04, 3.9879296E+04, 4.0099196E+04, 4.0052539E+04, 4.0009473E+04, 3.9967072E+04, 4.0033119E+04, 3.9986071E+04, 3.9995917E+04, 4.0939414E+04, 3.9982026E+04, 3.99855869E+04, 3.9955869E+04,	2.1965091E+07, 2.2000469E+07, 2.2033191E+07, 2.2059947E+07, 2.2054514E+07, 2.2037290E+07, 2.2034773E+07, 2.2161891E+07, 2.2139260E+07, 2.2119273E+07, 2.2141239E+07, 2.2141239E+07, 2.2129457E+07, 2.2129457E+07, 2.2162845E+07, 2.2135457E+07, 2.2183738E+07, 2.2074846E+07, 2.2047191E+07,

TABLE 6.2.1-27 (SHEET 22 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
.60011, .61007, .62001, .63011, .64010, .65010, .66002, .67003, .68012, .69009, .70008, .71006, .72007, .73009, .74006, .75003, .76008, .77027, .78014, .79004, .80006, .81003, .82008, .83008, .84013, .85007, .86008, .87003, .86008, .87003, .88008, .87003, .88008, .87003, .88008, .87003, .88008, .87003, .88008, .87003, .88008, .87003, .88008, .87003, .88008, .89009, .90005, .91010,	3.9801350E+04, 3.9806850E+04, 3.9806850E+04, 3.9785117E+04, 3.9745377E+04, 3.9683897E+04, 3.9627585E+04, 3.9582612E+04, 3.9535443E+04, 3.9446800E+04, 3.9348911E+04, 3.9348911E+04, 3.93266575E+04, 3.9221422E+04, 3.9101610E+04, 3.9101610E+04, 3.9101610E+04, 3.8954031E+04, 3.8954031E+04, 3.8880815E+04, 3.8880815E+04, 3.88703083E+04, 3.8703083E+04, 3.8743384E+04, 3.8743384E+04, 3.8743384E+04, 3.8743384E+04, 3.8793083E+04, 3.8698525E+04, 3.8596282E+04, 3.8596282E+04, 3.8596282E+04,	(Btu/s) 2.2054765E+07, 2.2066753E+07, 2.2068233E+07, 2.2061235E+07, 2.2044375E+07, 2.2015240E+07, 2.1989164E+07, 2.1969491E+07, 2.1948660E+07, 2.1929156E+07, 2.1929156E+07, 2.1887528E+07, 2.1887528E+07, 2.1849517E+07, 2.1849517E+07, 2.1831721E+07, 2.1756044E+07, 2.1756044E+07, 2.1720941E+07, 2.1720941E+07, 2.1649063E+07, 2.1649063E+07, 2.1566116E+07, 2.1569190E+07, 2.1569190E+07, 2.1569190E+07, 2.15607963E+07, 2.15607963E+07, 2.15607963E+07, 2.15607963E+07, 2.15607963E+07, 2.1550798E+07, 2.1550798E+07, 2.1522539E+07, 2.1484667E+07, 2.1431046E+07, 2.1431046E+07, 2.1414617E+07,
.92009, .93010, .94004, .95009, .96010, .97009, .98005, .99008,	3.8312887E+04, 3.8262021E+04, 3.8201289E+04, 3.8138982E+04, 3.8085254E+04, 3.8037122E+04, 3.7991015E+04, 3.7942798E+04,	2.1396098E+07, 2.1372841E+07, 2.1343908E+07, 2.1314158E+07, 2.1288159E+07, 2.1267278E+07, 2.1246553E+07, 2.1224593E+07,
.55000,	5.1572130L10 7 ,	Z. 1227000L 107,

TABLE 6.2.1-27 (SHEET 23 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) 1.00007, 1.05003, 1.10019, 1.15003, 1.25003, 1.25003, 1.35000, 1.40002, 1.45010, 1.50011, 1.55009, 1.60007, 1.65004, 1.70006, 1.75012, 1.80012, 1.85000, 1.90002, 1.95007, 2.00004, 2.15006, 2.25001, 2.30003, 2.35015, 2.40009, 2.45005, 2.50012, 2.55001, 2.60002, 2.65010, 2.70000, 2.75000, 2.80001,		
2.85006, 2.90003, 2.95005, 3.00012,	3.3611093E+04, 3.2983430E+04, 3.2431145E+04, 3.2121901E+04,	1.8996154E+07, 1.8657303E+07, 1.8159907E+07, 1.8199162E+07,

TABLE 6.2.1-27 (SHEET 24 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) .00000, .00101, .00202, .00301, .00401, .00501, .00601, .00701, .00804, .00902, .01003, .01101, .01202, .01301, .01402, .01500, .01600, .01704, .01804, .02904, .02106, .02106, .02201, .02301, .02402, .02500, .02604, .02703, .02801, .02903, .03005, .03102, .03202, .03304, .03401, .03504,		
.03605, .03703, .03800, .03901,	3.0786569E+04, 3.0631717E+04, 3.0466928E+04, 3.0358633E+04,	1.6934025E+07, 1.6847147E+07, 1.6756951E+07, 1.6698084E+07,

TABLE 6.2.1-27 (SHEET 25 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
.04000, .04102, .04201, .04301, .04400, .04502, .04600, .04700, .04801, .04902, .05000, .05101, .05200, .05301, .05402, .05502, .05602, .05703, .05804, .06902, .06103, .06203, .06301, .06401, .06502, .06600, .06702, .06807, .06906, .07004, .07103, .07203, .07203, .07305,	3.0438252E+04, 3.0846188E+04, 3.1253101E+04, 3.1106960E+04, 3.0710999E+04, 3.0570958E+04, 3.0688257E+04, 3.0701911E+04, 3.0574827E+04, 3.0832885E+04, 3.1189824E+04, 3.1912778E+04, 3.3570213E+04, 3.6325123E+04, 3.9631396E+04, 4.0664118E+04, 4.1526297E+04, 4.206121E+04, 4.2469172E+04, 4.2978961E+04, 4.2978961E+04, 4.2978961E+04, 4.3026975E+04, 4.3549054E+04, 4.4721753E+04, 4.4721753E+04, 4.4721753E+04, 4.47325363E+04, 4.8775747E+04, 4.9224183E+04, 4.9270169E+04, 4.8890924E+04,	(Btu/s) 1.6745592E+07, 1.6973687E+07, 1.7193824E+07, 1.7106070E+07, 1.6888803E+07, 1.68815748E+07, 1.6881790E+07, 1.6885873E+07, 1.68818120E+07, 1.6964301E+07, 1.7162053E+07, 1.7569750E+07, 1.7569750E+07, 2.1316538E+07, 2.1316538E+07, 2.1560199E+07, 2.1560199E+07, 2.1830694E+07, 2.2406067E+07, 2.2876487E+07, 2.3395779E+07, 2.3395779E+07, 2.3606824E+07, 2.3700508E+07, 2.4661505E+07, 2.5462312E+07, 2.6895365E+07, 2.7137644E+07, 2.6936090E+07,
.07400, .07500, .07606, .07704, .07806,	4.8385139E+04, 4.7968558E+04, 4.7563277E+04, 4.7314610E+04, 4.7036794E+04,	2.6654382E+07, 2.6391960E+07, 2.6202327E+07, 2.6064344E+07, 2.5908764E+07,

TABLE 6.2.1-27 (SHEET 26 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
.07904, .08000, .08106, .08202, .08303, .08402, .08504, .08606, .08701, .08803, .08901, .09005, .09101, .09203, .09300, .09402, .09504, .09504, .09602, .09704, .09803, .10006, .11508, .11508, .12511,	(lb/s) 4.6723180E+04, 4.6426196E+04, 4.6243806E+04, 4.6246547E+04, 4.6390383E+04, 4.6699180E+04, 4.6613903E+04, 4.6324981E+04, 4.5908154E+04, 4.5514602E+04, 4.55170201E+04, 4.4550294E+04, 4.4550294E+04, 4.4258533E+04, 4.4041145E+04, 4.3883324E+04, 4.3748654E+04, 4.3731915E+04, 4.495327E+04, 4.4993727E+04, 4.4915203E+04, 4.5843578E+04, 4.7961126E+04, 4.8641610E+04, 4.8600261E+04,	(Btu/s) 2.5733981E+07, 2.5570341E+07, 2.5472215E+07, 2.5476682E+07, 2.5558327E+07, 2.5665636E+07, 2.5726088E+07, 2.5674637E+07, 2.5511111E+07, 2.5279153E+07, 2.5062319E+07, 2.4872751E+07, 2.4710995E+07, 2.4710995E+07, 2.44529427E+07, 2.4250486E+07, 2.4369254E+07, 2.4089902E+07, 2.4089902E+07, 2.4089902E+07, 2.470897E+07, 2.4568299E+07, 2.4738296E+07, 2.4738296E+07, 2.641304E+07, 2.6811261E+07, 2.6791935E+07,
.12311, .13003, .13507, .14007, .14510, .15004,	4.8897292E+04, 4.7822446E+04, 4.6760026E+04, 4.6613100E+04, 4.5690755E+04,	2.6954703E+07, 2.6954703E+07, 2.6353890E+07, 2.5769811E+07, 2.5690052E+07, 2.5177113E+07,
.15004, .15504, .16009, .16521, .17003, .17512, .18005, .18515,	4.5690755E+04, 4.5251619E+04, 4.5889095E+04, 4.6090261E+04, 4.6472870E+04, 4.7251129E+04, 4.7225150E+04, 4.6856876E+04,	2.5177113E+07, 2.4940772E+07, 2.5299141E+07, 2.5411637E+07, 2.5629872E+07, 2.6064692E+07, 2.6048738E+07, 2.5847037E+07,
.19008, .19500,	4.6705900E+04, 4.6273001E+04,	2.5764932E+07, 2.5525670E+07,

TABLE 6.2.1-27 (SHEET 27 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) .20008, .21005, .22011, .23005, .24013, .25001, .26009, .27006, .28006, .29009, .30002, .31009, .32012, .33009, .34001, .35007, .36007, .37007, .38005, .39016, .44008, .41016, .42012, .43008, .44004, .45003, .47005, .48007, .49020, .50011, .51002, .52003, .53004, .54004, .55004, .55004, .55004, .56007, .57003, .58003, .59001,		
.60015,	4.5761354E+04,	2.5550060E+07,

TABLE 6.2.1-27 (SHEET 28 OF 36)

(436-in.² BREAK)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) .61002, .62006, .63001, .64010, .65006, .66004, .67008, .68008, .69010, .70013, .71008, .72015, .73015, .74007, .75000, .76013, .77005, .78003, .79011, .80002, .81009, .82019, .83017, .84015, .85012, .86011, .87001, .88002, .89016, .90029, .91012, .92001, .93007, .94009, .95014,		
.96017, .97012, .98005, .99007,	4.2928460E+04, 4.2855367E+04, 4.2795246E+04, 4.2780821E+04,	2.4254898E+07, 2.4218973E+07, 2.4190366E+07, 2.4187831E+07,

TABLE 6.2.1-27 (SHEET 29 OF 36)

(436-in.² BREAK)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
1.00003, 1.05004, 1.10007, 1.15004, 1.20025, 1.25006, 1.30021, 1.35002, 1.40027, 1.45004, 1.50000, 1.55013, 1.60003, 1.65008, 1.70000, 1.75006, 1.80008, 1.85002, 1.90002, 1.95014, 2.00003, 2.10007, 2.15010, 2.25009, 2.35003, 2.40001, 2.45017, 2.55005, 2.55005, 2.65026,	(lb/s) 4.2752246E+04, 4.2539695E+04, 4.2334300E+04, 4.2209823E+04, 4.2149730E+04, 4.2092436E+04, 4.2110966E+04, 4.2178652E+04, 4.1966932E+04, 4.1784741E+04, 4.1699789E+04, 4.1435150E+04, 4.1435150E+04, 4.1483172E+04, 4.1483365E+04, 4.1483365E+04, 4.1016295E+04, 4.1016295E+04, 4.0870107E+04, 4.0455307E+04, 3.9934443E+04, 3.9753225E+04, 3.9934443E+04, 3.9753225E+04, 3.993013E+04, 3.8805140E+04, 3.8805140E+04, 3.7919102E+04, 3.7768094E+04, 3.7768094E+04, 3.7768094E+04, 3.7768094E+04, 3.7325121E+04, 3.6306581E+04, 3.6306581E+04, 3.5600723E+04, 3.4826145E+04,	(Btu/s) 2.4177108E+07, 2.4082305E+07, 2.3988517E+07, 2.3939257E+07, 2.3926104E+07, 2.3914261E+07, 2.3946258E+07, 2.3949385E+07, 2.3905238E+07, 2.3801830E+07, 2.3801830E+07, 2.3735546E+07, 2.3722962E+07, 2.3754789E+07, 2.3754789E+07, 2.3754789E+07, 2.37550974E+07, 2.3550974E+07, 2.3550974E+07, 2.3550974E+07, 2.3295283E+07, 2.2954260E+07, 2.2718398E+07, 2.2718398E+07, 2.2466812E+07, 2.2466812E+07, 2.2466812E+07, 2.2524496E+07, 2.2524496E+07, 2.2524496E+07, 2.2524496E+07, 2.2052489E+07, 2.1557137E+07, 2.1557137E+07, 2.1928827E+07, 2.0928827E+07, 2.0928827E+07, 2.0513480E+07,
2.05026, 2.70004, 2.75012, 2.80013, 2.85005, 2.90000, 3.00007,	3.4058488E+04, 3.3204439E+04, 3.2227744E+04, 3.1113386E+04, 3.0258151E+04, 3.0030674E+04, 2.9782520E+04,	2.0313480E+07, 2.0100456E+07, 1.9641103E+07, 1.9115998E+07, 1.8509041E+07, 1.8052239E+07, 1.7966679E+07, 1.7874253E+07,

TABLE 6.2.1-27 (SHEET 30 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow
<u>11111C (3)</u>	<u> </u>	<u>(Dtars)</u>
Time (s) .00000, .00102, .00201, .00303, .00403, .00500, .00602, .00702, .00800, .00901, .01003, .01104, .01202, .01301, .01400, .01503, .01601, .01701, .01801, .01903, .02102, .02300, .02403, .02504, .02601, .02702, .02802, .02904, .03000, .03100, .03201, .03302, .03402, .03501, .0360	0. 7.7823700E+03, 1.1086818E+04, 2.2100842E+04, 3.4238857E+04, 4.1741684E+04, 4.6481946E+04, 4.9110254E+04, 5.0377887E+04, 5.0849559E+04, 5.0761880E+04, 5.0769941E+04, 5.1006142E+04, 5.1391505E+04, 5.1391505E+04, 5.2309698E+04, 5.2748453E+04, 5.3411771E+04, 5.3587249E+04, 5.3662466E+04, 5.3650130E+04, 5.3650130E+04, 5.365954E+04, 5.3436329E+04, 5.3436329E+04, 5.3436329E+04, 5.3436329E+04, 5.3436329E+04, 5.349493E+04, 5.3029110E+04, 5.3036867E+04, 5.3029110E+04, 5.30331104E+04, 5.3549493E+04, 5.3794178E+04, 5.3794178E+04, 5.3794178E+04, 5.3794178E+04,	(Btu/s) 0. 4.9504728E+06, 7.0489849E+06, 1.4051387E+07, 2.1774422E+07, 2.6547063E+07, 3.29556339E+07, 3.2012324E+07, 3.2301064E+07, 3.2311861E+07, 3.234808E+07, 3.2192830E+07, 3.2242396E+07, 3.2942396E+07, 3.2942396E+07, 3.394472E+07, 3.3923221E+07, 3.39439E+07, 3.3911858E+07, 3.4063267E+07, 3.4063267E+07, 3.3995967E+07, 3.3912156E+07, 3.3995967E+07, 3.3912156E+07, 3.3912156E+07, 3.392261E+07, 3.3665533E+07, 3.3665533E+07, 3.3665533E+07, 3.3748506E+07, 3.3864763E+07, 3.3748506E+07, 3.3864763E+07, 3.406787E+07, 3.4165108E+07, 3.4337820E+07,
.03702,	5.4337548E+04,	3.4515029E+07,

TABLE 6.2.1-27 (SHEET 31 OF 36)

	Mass Flow	Energy Flow
Time (s)	<u>(lb/s)</u>	(Btu/s)
.03801,	5.4603859E+04,	3.4686046E+07,
.03903,	5.4871375E+04,	3.4857557E+07,
.04002,	5.5115012E+04,	3.5013532E+07,
.04104,	5.5345808E+04,	3.5161050E+07,
.04203,	5.5556895E+04,	3.5295796E+07,
.04302,	5.5742463E+04,	3.5414074E+07,
.04402,	5.5914907E+04,	3.5523738E+07,
.04500,	5.6061661E+04,	3.5616908E+07,
.04602,	5.6195171E+04,	3.5701406E+07, 3.5770270E+07,
.04701, .04800,	5.6304293E+04, 5.6400251E+04,	3.5830544E+07,
.04000,	5.6479238E+04,	3.5879874E+07,
.05003,	5.6538079E+04,	3.5916299E+07,
.05101,	5.6580392E+04,	3.5942104E+07,
.05202,	5.6607447E+04,	3.5958098E+07,
.05301,	5.6618964E+04,	3.5964131E+07,
.05403,	5.6614906E+04,	3.5960166E+07,
.05507,	5.6595249E+04,	3.5948240E+07,
.05604,	5.6563240E+04,	3.5924545E+07,
.05706,	5.6517173E+04,	3.5893887E+07,
.05808,	5.6459413E+04,	3.5855845E+07,
.05903,	5.6395002E+04,	3.5813728E+07,
.06000,	5.6323849E+04,	3.5767426E+07,
.06105,	5.6238056E+04,	3.5711826E+07,
.06204, .06303,	5.6152107E+04, 5.6061848E+04,	3.5656354E+07, 3.5598461E+07,
.06406,	5.5969187E+04,	3.5539262E+07,
.06503,	5.5882016E+04,	3.5483752E+07,
.06600,	5.5797599E+04,	3.5430181E+07,
.06705,	5.5711644E+04,	3.5375809E+07,
.06803,	5.5631219E+04,	3.5325235E+07,
.06907 [°] ,	5.5559086E+04,	3.5280130E+07,
.07002,	5.5492411E+04,	3.5238611E+07,
.07103,	5.5432132E+04,	3.5201303E+07,
.07201,	5.5378454E+04,	3.5168222E+07,
.07301,	5.5330161E+04,	3.5138690E+07,
.07402,	5.5287428E+04,	3.5112743E+07,
.07501, .07601,	5.5249761E+04, 5.5215621E+04,	3.5090042E+07, 3.5069674E+07,
.07001,	5.5213621E+04, 5.5185947E+04,	3.5052144E+07,
.07701,	0.01000 1 1 L 10 1 ,	0.0002 177L · 01,

TABLE 6.2.1-27 (SHEET 32 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
.07800, .07900, .08000, .08100, .08201, .08301, .08402, .08501, .08600, .08701, .08803, .08902, .09001, .09100, .09201, .09303, .09402, .09503, .09402, .09503, .09601, .09700, .09802, .10505, .11004, .11506, .12000, .12507, .13006, .13502, .14002,	(lb/s) 5.5515956E+04, 5.5135988E+04, 5.5114523E+04, 5.5095190E+04, 5.5077046E+04, 5.5059845E+04, 5.5042363E+04, 5.5024404E+04, 5.5005332E+04, 5.4984427E+04, 5.4937813E+04, 5.4912757E+04, 5.4887504E+04, 5.4889475E+04, 5.4818668E+04, 5.4818668E+04, 5.476706E+04, 5.476772E+04, 5.4768772E+04, 5.4762804E+04, 5.4762804E+04, 5.4762804E+04, 5.4757413E+04, 5.47634156E+04, 5.2981994E+04, 5.2981994E+04, 5.2125082E+04, 5.1385350E+04, 5.0623676E+04, 4.9952734E+04, 4.9952734E+04,	(Btu/s) 3.5036721E+07, 3.5023107E+07, 3.5010852E+07, 3.4999922E+07, 3.4989734E+07, 3.4980098E+07, 3.4970280E+07, 3.4960147E+07, 3.4949397E+07, 3.4924813E+07, 3.4911638E+07, 3.4897802E+07, 3.4884017E+07, 3.4870828E+07, 3.4889563E+07, 3.4839804E+07, 3.4839804E+07, 3.4829916E+07, 3.4825796E+07, 3.4825796E+07, 3.4756575E+07, 3.4334414E+07, 3.3711323E+07, 3.3711323E+07, 3.2700539E+07, 3.2700539E+07, 3.1794294E+07, 3.1590996E+07,
.14507, .15002, .15504,	4.9659208E+04, 4.9826967E+04, 4.9958804E+04.	3.1619886E+07, 3.1730674E+07, 3.1816492E+07,
.15002, .15504, .16014, .16512, .17007, .17500, .18002, .18507,	4.9826967E+04, 4.9958804E+04, 4.9974844E+04, 4.9873952E+04, 4.9676572E+04, 4.9424401E+04, 4.9169537E+04, 4.8977379E+04,	3.1730674E+07, 3.1816492E+07, 3.1827172E+07, 3.1762947E+07, 3.1637817E+07, 3.1478882E+07, 3.1319320E+07, 3.1200412E+07,
.19008,	4.8891947E+04,	3.1149864E+07,

TABLE 6.2.1-27 (SHEET 33 OF 36)

	Mass Flow	Energy Flow
Time (s)	(lb/s)	(Btu/s)
10512	4 900E220E±04	2 1162296E±07
.19512, .20017,	4.8905230E+04, 4.8971499E+04,	3.1162286E+07, 3.1208073E+07,
.21011,	4.9132117E+04,	3.1200073E+07, 3.1316312E+07,
.22004,	4.9118318E+04,	3.1311984E+07,
.23006,	4.8895594E+04,	3.1176020E+07,
.24001,	4.8547872E+04,	3.0963726E+07,
.25009,	4.8202471E+04,	3.0756576E+07,
.26004,	4.7929553E+04,	3.0596316E+07,
.27003,	4.7702439E+04,	3.0462393E+07,
.28009,	4.7508889E+04,	3.0345842E+07,
.29002,	4.7371087E+04,	3.0261785E+07,
.30001,	4.7306930E+04,	3.0223088E+07,
.31008,	4.7280472E+04,	3.0207649E+07,
.32012,	4.7230590E+04,	3.0177807E+07,
.33006,	4.7094913E+04,	3.0094690E+07,
.34005,	4.6887690E+04,	2.9968310E+07,
.35010,	4.6650874E+04,	2.9824836E+07,
.36013,	4.6424525E+04,	2.9688533E+07,
.37004,	4.6213540E+04,	2.9562593E+07,
.38012,	4.6028987E+04,	2.9452992E+07,
.39005,	4.5905610E+04,	2.9381484E+07,
.40008,	4.5854401E+04,	2.9355651E+07,
.41013,	4.5863015E+04,	2.9367338E+07,
.42003,	4.5897394E+04,	2.9394808E+07,
.43010,	4.5915956E+04,	2.9411982E+07,
.44013,	4.5895521E+04,	2.9403662E+07,
.45009,	4.5822802E+04,	2.9361067E+07,
.46003,	4.5702115E+04,	2.9287550E+07,
.47007,	4.5560837E+04,	2.9201025E+07,
.48012,	4.5440327E+04,	2.9128036E+07,
.49005,	4.5370502E+04,	2.9087580E+07,
.50001,	4.5356383E+04,	2.9082777E+07,
.51009,	4.5385682E+04,	2.9105652E+07,
.52009,	4.5432669E+04,	2.9139564E+07,
.53005,	4.5468619E+04,	2.9166079E+07,
.54006,	4.5468395E+04,	2.9169117E+07,
.55003,	4.5416172E+04,	2.9138892E+07,
.56000, 57006	4.5321254E+04,	2.9081899E+07, 2.9015687E+07,
.57006, 58001	4.5210656E+04, 4.5115650E+04,	
.58001,	4.5115030ET04,	2.8960172E+07,

TABLE 6.2.1-27 (SHEET 34 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
Time (s) .59004, .60012, .61010, .62013, .63009, .64014, .65011, .66013, .67009, .68003, .69012, .70008, .71002, .72000, .73002, .74007, .75002, .76001, .77004, .78001, .79005, .80009, .81002, .82009, .83004, .84013,		
.84013, .85007, .86005, .87006, .88001, .89009, .90003, .91005, .92010,	4.4306106E+04, 4.4261121E+04, 4.4201427E+04, 4.4147923E+04, 4.4100598E+04, 4.4061089E+04, 4.4041189E+04, 4.4031122E+04, 4.4021949E+04, 4.4004878E+04,	2.8662665E+07, 2.8644991E+07, 2.8618225E+07, 2.8595735E+07, 2.8577514E+07, 2.8564640E+07, 2.8564429E+07, 2.8570789E+07, 2.8577993E+07, 2.8580150E+07,
.94003, .95000, .96007, .97011, .98001, .99004,	4.3975310E+04, 4.3932837E+04, 4.3877626E+04, 4.3813505E+04, 4.3746579E+04, 4.3679310E+04,	2.8574674E+07, 2.8561112E+07, 2.8539750E+07, 2.8512862E+07, 2.8434177E+07, 2.8455659E+07,

TABLE 6.2.1-27 (SHEET 35 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
1.00009,	4.3616582E+04,	2.8430269E+07,
1.05004,	4.3362851E+04,	2.8343817E+07,
1.10009,	4.3106996E+04,	2.8203634E+07,
1.15007,	4.2656312E+04,	2.8059859E+07,
1.20002,	4.2276635E+04,	2.7914949E+07,
1.25002,	4.1996846E+04,	2.7844326E+07,
1.30006,	4.1778160E+04,	2.7808761E+07,
1.35006,	4.1502658E+04,	2.7729192E+07,
1.40003,	4.1145142E+04,	2.7590021E+07,
1.45007,	4.0719791E+04,	2.7403645E+07,
1.50011,	4.0278464E+04,	2.7196215E+07,
1.55009,	3.9836748E+04,	2.6978928E+07,
1.60001, 1.65013	3.9415183E+04,	2.6769679E+07,
1.65012, 1.70012,	3.8998375E+04, 3.8602806E+04,	2.6559661E+07, 2.6359988E+07,
1.75012,	3.8221231E+04,	2.6165738E+07,
1.80006,	3.7854297E+04,	2.5973809E+07,
1.85007,	3.7502521E+04,	2.5782014E+07,
1.90000,	3.7155071E+04,	2.5583127E+07,
1.95010,	3.6808801E+04,	2.5377270E+07,
2.00009,	3.6453550E+04,	2.5162290E+07,
2.05014,	3.6085701E+04,	2.4941049E+07,
2.10011,	3.5697676E+04,	2.4713441E+07,
2.15005,	3.5290806E+04,	2.4482273E+07,
2.20000,	3.4873172E+04,	2.4253393E+07,
2.25004,	3.4442934E+04,	2.4025041E+07,
2.30012,	3.4020636E+04,	2.3801494E+07,
2.35007,	3.3630302E+04,	2.3589986E+07,
2.40002,	3.3267885E+04,	2.3389694E+07,
2.45004,	3.2900597E+04,	2.3195920E+07,
2.50007,	3.2674711E+04,	2.3007341E+07,
2.55013,	3.2398991E+04,	2.2818065E+07,
2.60007,	3.2112088E+04,	2.2622375E+07,
2.65003, 2.70013,	3.1797713E+04, 3.1441037E+04,	2.2417317E+07, 2.2198094E+07,
2.75000, 2.75000,	3.1057741E+04,	2.2196094E+07, 2.1971109E+07,
2.80009,	3.0669535E+04,	2.1741814E+07,
2.85004,	3.0303664E+04,	2.1519357E+07,
2.00004,	0.000000TL · 0T,	2.1010007 L 107,

TABLE 6.2.1-27 (SHEET 36 OF 36)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)
2.90009,	2.9952884E+04,	2.1304776E+07,
2.95003,	2.9588947E+04,	2.1034229E+07,
3.00005,	2.9263868E+04,	2.0871117E+07,

TABLE 6.2.1-28 (SHEET 1 OF 7)

PRESSURIZER COMPARTMENT MASS AND ENERGY RELEASE PRESSURIZER SURGE LINE BREAK

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)	Average Enthalpy (Btu/lb)
0.00000 0.00251 0.00501 0.00752 0.01002 0.01250 0.01501 0.01754 0.02002 0.02250 0.02505 0.02752 0.03001 0.03258 0.03503 0.03750 0.04009 0.04259 0.04512 0.04761 0.05009 0.05264 0.05505 0.05751 0.06008 0.06255 0.06512 0.06759 0.07002 0.07250 0.07250 0.07250 0.07507 0.07754 0.08008 0.08255 0.08504 0.08755 0.09006 0.09261 0.09501 0.09751 0.10011	0.00 5.6681148E+04 1.6556361E+04 1.6699069E+04 1.9033506E+04 2.2089365E+04 2.1648161E+04 2.1247911E+04 2.0465838E+04 2.0393611E+04 2.0706044E+04 2.0991729E+04 2.0998600E+04 2.0997919E+04 2.1019840E+04 2.1019840E+04 2.1156624E+04 2.1156624E+04 2.1160405E+04 2.1095761E+04 2.1095761E+04 2.1085426E+04 2.1085426E+04 2.1085426E+04 2.0863697E+04 2.0863697E+04 2.0407265E+04 2.0407265E+04 2.0418448E+04 2.0410939E+04 2.0410939E+04 2.0410939E+04 2.0410939E+04 2.0410939E+04 2.0410939E+04 2.0305825E+04 2.0203846E+04 2.0124810E+04 2.0067276E+04 2.0091047E+04 2.0091047E+04 2.0099095E+04	0.00 1.1296008E+07 1.1212058E+07 1.1302997E+07 1.2830006E+07 1.4828262E+07 1.4533929E+07 1.4268193E+07 1.3752132E+07 1.3704347E+07 1.3907231E+07 1.4053966E+07 1.4096217E+07 1.4074876E+07 1.4107187E+07 1.4107187E+07 1.4197306E+07 1.4195514E+07 1.4156324E+07 1.4156324E+07 1.416331E+07 1.416331E+07 1.4089897E+07 1.3888722E+07 1.3767663E+07 1.370589E+07 1.370589E+07 1.3774417E+07 1.3774417E+07 1.3775092E+07 1.3753315E+07 1.3753315E+07 1.3753315E+07 1.3753315E+07 1.3753315E+07 1.3753315E+07 1.376990E+07 1.366573E+07 1.3566573E+07 1.3566573E+07 1.356657427E+07	0.00 677.17 677.21 676.86 674.07 671.29 671.51 671.96 671.99 671.65 671.20 671.20 671.20 671.20 671.20 671.20 671.20 671.20 671.30 671.31 670.95 670.95 670.95 670.95 670.95 671.28 671.36 671.28 671.28 671.28 671.28 671.28 671.28 671.28 671.28 671.28 671.28 671.28 671.27 671.28 671.41 671.59 671.49
0.10011	2.0190093E+04	1.3337427 = +07	071.49

TABLE 6.2.1-28 (SHEET 2 OF 7)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)	Average Enthalpy <u>(Btu/lb)</u>
		1.3644022E+07 1.3741182E+07 1.3834228E+07 1.3889843E+07 1.3896965E+07 1.3855823E+07 1.3687301E+07 1.3687301E+07 1.3584155E+07 1.3479844E+07 1.3247406E+07 1.3247406E+07 1.2988402E+07 1.2988402E+07 1.2682997E+07 1.2682997E+07 1.2568006E+07 1.2531659E+07 1.2531659E+07 1.2494497E+07 1.2482868E+07 1.2494497E+07 1.2482868E+07 1.2508250E+07 1.219261E+07 1.12993E+07 1.1687137E+07 1.1687159E+07 1.1657900E+07 1.1599461E+07 1.1599461E+07 1.1571232E+07	
0.20257 0.20515 0.20764 0.21013	1.7114304E+04 1.7075604E+04 1.7047204E+04 1.7030892E+04	1.1545858E+07 1.1520650E+07 1.1502146E+07 1.1491517E+07	674.63 674.68 674.72 674.75

TABLE 6.2.1-28 (SHEET 3 OF 7)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)	Average Enthalpy (Btu/lb)
0.21262 0.21509 0.21755 0.22014 0.22261 0.22518 0.22757 0.23002 0.23261 0.23511 0.23756 0.24007 0.24253 0.24509 0.24752 0.25009 0.25256 0.25505 0.25751 0.26005 0.26251 0.26504 0.26751 0.26005 0.27751 0.27009 0.27265 0.277509 0.27751 0.28003 0.28253 0.28512 0.28755 0.29509 0.29256 0.29509 0.29256 0.29509 0.30265 0.30517 0.300753 0.31006	1.7026041E+04 1.7031337E+04 1.7044630E+04 1.7064281E+04 1.7085144E+04 1.7120939E+04 1.7133999E+04 1.7135943E+04 1.7121402E+04 1.7091493E+04 1.7091493E+04 1.6981075E+04 1.6981075E+04 1.6633374E+04 1.6633374E+04 1.6466594E+04 1.6403107E+04 1.6352876E+04 1.625893E+04 1.632701E+04 1.6127011E+04 1.6127011E+04 1.5898962E+04 1.5898962E+04 1.5898962E+04 1.5808551E+04 1.5808165E+04 1.5808165E+04 1.5803688E+04 1.5753769D+04 1.5753769D+04 1.5744724E+04	1.1488355E+07 1.1491802E+07 1.1500441E+07 1.1513204E+07 1.1526752E+07 1.1539031E+07 1.1549913E+07 1.1558321E+07 1.1559323E+07 1.1559323E+07 1.1549700E+07 1.1530055E+07 1.1457784E+07 1.1457784E+07 1.1457784E+07 1.14207E+07 1.1230921E+07 1.1171824E+07 1.1171824E+07 1.1048104E+07 1.1027787E+07 1.1048104E+07 1.098473E+07 1.0984763E+07 1.0984763E+07 1.0984763E+07 1.0984763E+07 1.0986874E+07 1.0938740E+07 1.0938740E+07 1.0752073E+07 1.0709102E+07 1.0709102E+07 1.0709102E+07 1.0709102E+07 1.0709102E+07 1.0709102E+07 1.0693108E+07 1.0692807E+07 1.0657243E+07 1.0657243E+07 1.0651197E+07 1.0656385E+07	674.75 674.74 674.70 674.67 674.64 674.61 674.58 674.57 674.58 674.61 674.66 674.74 674.83 674.95 675.07 675.20 675.33 675.44 675.53 675.61 675.65 675.71 675.74 675.78 675.78 675.74 675.78 675.74 675.78 675.74 675.78 676.16 676.28 676.38 676.38 676.41 676.41 676.49 676.49
0.31256 0.31501 0.31767 0.32014	1.5753012E+04 1.5760820E+04 1.5737615E+04 1.5735895E+04	1.0661304E+07 1.0646068E+07 1.0644859E+07	676.47 676.44 676.47 676.47

TABLE 6.2.1-28 (SHEET 4 OF 7)

Time (s)	Mass Flow _(lb/s)_	Energy Flow (Btu/s)	Average Enthalpy <u>(Btu/lb)</u>
		Energy Flow (Btu/s) 1.0643956E+07 1.0642618E+07 1.0640794E+07 1.0638378E+07 1.0636121E+07 1.0630912E+07 1.0628319E+07 1.0625820E+07 1.0623609E+07 1.0621254E+07 1.0619096E+07 1.0617177E+07 1.0615038E+07 1.0615038E+07 1.0607961E+07 1.0602767E+07 1.0599906E+07 1.0599906E+07 1.05993197E+07 1.0593197E+07 1.0587321E+07 1.0579890E+07 1.0577652E+07 1.0577652E+07 1.05756689E+07 1.0575927E+07 1.0573960E+07	Enthalpy
0.41775 0.42001 0.42268 0.42508 0.42781 0.43027	1.4532551E+04 1.5632366E+04 1.5632315E+04 1.5632378E+04 1.5632537E+04 1.5632708E+04	1.0573584E+07 1.0573381E+07 1.0573254E+07 1.0573212E+07 1.0573215E+07 1.0573240E+07	676.38 676.38 676.37 676.37 676.36 676.35

TABLE 6.2.1-28 (SHEET 5 OF 7)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)	Average Enthalpy (Btu/lb)
0.64028 0.65009 0.66010	1.5529494E+04 1.5523642E+04 1.5516674E+04	1.0496323E+07 1.0491943E+07 1.0486820E+07	675.90 675.87 675.84

TABLE 6.2.1-28 (SHEET 6 OF 7)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)	Average Enthalpy <u>(Btu/lb)</u>
1.40047 1.45008 1.50052	1.5158958E+04 1.5135654E+04 1.5108628E+04	1.0214058E+07 1.0196263E+07 1.0175997E+07	673.80 673.66 673.52

TABLE 6.2.1-28 (SHEET 7 OF 7)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)	Average Enthalpy (Btu/lb)
1.55020	1.5082453E+04	1.0156373E+07	673.39
1.60001	1.5056220E+04	1.0136713E+07	673.26
1.65041	1.5029192E+04	1.0116506E+07	673.12
1.70057	1.5003251E+04	1.0097044E+07	672.99
1.75042	1.4978790E+04	1.0078568E+07	672.86
1.80008	1.4954574E+04	1.0060259E+07	672.72
1.85001	1.4910178E+04	1.0027829E+07	672.55
1.90030	1.4937484E+04	1.0045448E+07	672.50
1.95350	1.4846071E+04	9.9803806E+06	672.26
2.00022	1.4821801E+04	9.9621501E+06	672.13

TABLE 6.2.1-28A (SHEET 1 OF 7)

PRESSURIZER COMPARTMENT MASS AND ENERGY RELEASE PRESSURIZER SPRAY LINE BREAK

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)	Average Enthalpy (Btu/lb)
0.00000 0.00251 0.00502 0.00751 0.01002 0.01251 0.01502 0.01755 0.02003 0.02255 0.02505 0.02754 0.03004 0.03259 0.03507 0.03753 0.04002 0.04251 0.04763 0.05005 0.05263 0.05263 0.05518 0.05769 0.06510 0.06750 0.07250 0.07250 0.07765 0.08003 0.07250 0.07765 0.08003 0.07765 0.08003 0.07765 0.08003 0.08254 0.08757 0.09001 0.09257 0.09509 0.09759 0.10002	0.00 5.5520269E+03 5.7566695E+03 5.6923083E+03 5.6156477E+03 5.5820416E+03 5.6059056E+03 6.0469291E+03 6.0807799E+03 6.0942687E+03 6.3222326E+03 6.3772227E+03 6.4620881E+03 6.5151813E+03 6.5151813E+03 6.5151813E+03 6.5151813E+03 6.5151813E+03 6.5151813E+03 6.5151813E+03 6.5151813E+03 6.3427181E+03 6.2773775E+03 6.2551735E+03 6.2551735E+03 6.2524046E+03 6.2668754E+03 6.3249883E+03 6.3249883E+03 6.3249883E+03 6.3249883E+03 6.3276385E+03 6.3276385E+03 6.3455081E+03 6.3655132E+03 6.3688606E+03 6.3655132E+03 6.3688606E+03 6.3688606E+03 6.3679E+03 6.36708749E+03 6.0708749E+03	0.00 3.4074591E+06 3.5214197E+06 3.4814317E+06 3.4348228E+06 3.4132128E+06 3.4246446E+06 3.6028096E+06 3.6722760E+06 3.6897530E+06 3.6961327E+06 3.7737113E+06 3.8554807E+06 3.9026378E+06 3.9026378E+06 3.9088990E+06 3.9305146E+06 3.9287741E+06 3.8870628E+06 3.7899141E+06 3.7741758E+06 3.7741758E+06 3.7741758E+06 3.7818297E+06 3.7818297E+06 3.8139264E+06 3.8139264E+06 3.8139641E+06 3.7306914E+06 3.7306914E+06 3.7796317E+06	0.00 613.73 611.71 611.60 611.65 611.46 610.90 608.41 607.30 606.79 606.49 605.52 604.57 603.93 603.66 603.29 603.22 603.54 603.74 603.64 603.22 602.80 602.80 602.80 602.80 602.44 602.37 602.63 602.63 602.92 603.31 603.63 603.63 603.63 602.92 603.31 603.63 603.63 603.77 603.63 602.63 602.93 603.63 603.63 603.63
0.10257	6.3664980E+03	3.8325765E+06	601.99

TABLE 6.2.1-28A (SHEET 2 OF 7)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)	Average Enthalpy (Btu/lb)
0.20509 0.20760 0.21005 0.21354	6.0330499E+03 6.0241617E+03 6.0191035E+03 6.0197127E+03	3.6404113E+06 3.6353420E+06 3.6324522E+06 3.6327939E+06	603.41 603.46 603.49 603.48

TABLE 6.2.1-28A (SHEET 3 OF 7)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)	Average Enthalpy (Btu/lb)
		3.6362513E+06 3.6419285E+06 3.6489364E+06 3.6571731E+06 3.6642500E+06 3.6708838E+06 3.6757012E+06 3.6785898E+06 3.6791040E+06 3.6791040E+06 3.6643848E+06 3.6536132E+06 3.6320047E+06 3.6320047E+06 3.6158252E+06 3.6179873E+06 3.6179873E+06 3.63237539E+06 3.6427624E+06 3.6528848E+06 3.6620230E+06 3.6620230E+06 3.6708478E+06 3.6682550E+06 3.6708478E+06 3.6325934E+06 3.6325934E+06 3.6325934E+06 3.6325934E+06 3.6325934E+06 3.6325934E+06 3.6325934E+06 3.6325934E+06 3.6585621E+06 3.6325934E+06 3.6585621E+06	
0.31753 0.32004 0.32254 0.32505	5.8599636E+03 5.8608011E+03 5.8629988E+03 5.8656482E+03	3.5417959E+06 3.5422668E+06 3.5435071E+06 3.5450037E+06	604.41 604.40 604.38 604.37

TABLE 6.2.1-28A (SHEET 4 OF 7)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)	Average Enthalpy (Btu/lb)
		3.5466905E+06 3.5482967E+06 3.5498034E+06 3.5514986E+06 3.5532429E+06 3.5554426E+06 3.5578797E+06 3.5604779E+06 3.5630314E+06 3.5668279E+06 3.5668906E+06 3.5633740E+06 3.5539585E+06 3.5473402E+06 3.5473402E+06 3.5292171E+06 3.5234085E+06 3.5234795E+06 3.5234795E+06 3.53539585E+06 3.5234795E+06 3.55358907E+06 3.55358907E+06 3.55358907E+06 3.55358907E+06 3.55358907E+06 3.55585146E+06 3.5720883E+06 3.5720883E+06 3.5720883E+06 3.5795690E+06 3.5795690E+06 3.5795690E+06 3.5795690E+06 3.579700E+06 3.579700E+06 3.579700E+06 3.579700E+06 3.5708002E+06 3.5567264E+06 3.5567264E+06 3.5567264E+06 3.5567264E+06 3.5567264E+06	
0.42776 0.43002 0.43273 0.43505	5.8596323E+03 5.8458384E+03 5.8331063E+03 5.8221311E+03	3.5409490E+06 3.5330901E+06 3.5258467E+06 3.5196022E+06	604.30 604.38 604.45 604.52

TABLE 6.2.1-28A (SHEET 5 OF 7)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)	Average Enthalpy (Btu/lb)
(s) 0.43751 0.44011 0.44254 0.44254 0.44512 0.44506 0.45255 0.45501 0.45755 0.46008 0.46255 0.46505 0.46756 0.47006 0.47253 0.47514 0.47751 0.48017 0.48257 0.48509 0.48757 0.49011 0.49257 0.49505 0.49760 0.50011 0.51009 0.52009 0.53006 0.54005 0.55000 0.56006 0.57002 0.58007 0.59036 0.60004	(lb/s) 5.8130794E+03 5.8051154E+03 5.7994338E+03 5.7994338E+03 5.7908671E+03 5.7884098E+03 5.7868114E+03 5.7869305E+03 5.7865023E+03 5.7865023E+03 5.7891053E+03 5.7926250E+03 5.7940101E+03 5.7949974E+03 5.7945401E+03 5.7952080E+03 5.7945401E+03 5.7927826E+03 5.7945401E+03 5.7968716E+03 5.7768716E+03 5.7768716E+03 5.77697954E+03 5.7603602E+03 5.7603602E+03 5.7609480E+03 5.7844690E+03 5.7987721E+03 5.7987721E+03 5.7970116E+03 5.7987721E+03 5.7987721E+03 5.7987721E+03 5.7987721E+03 5.7987721E+03 5.79631540E+03 5.7631540E+03 5.7631540E+03 5.7548799E+03 5.7548799E+03 5.7548799E+03 5.7492097E+03	3.5144558E+06 3.5099211E+06 3.5066831E+06 3.5037894E+06 3.5017915E+06 3.5003828E+06 3.4994613E+06 3.4999035E+06 3.4998362E+06 3.4998863E+06 3.5006850E+06 3.5016617E+06 3.5026427E+06 3.5034098E+06 3.5039478E+06 3.5036413E+06 3.5036413E+06 3.4989748E+06 3.4989748E+06 3.4989748E+06 3.4989748E+06 3.4989748E+06 3.49834807E+06 3.49834807E+06 3.49834807E+06 3.4974703E+06 3.4841862E+06 3.4974703E+06 3.4974703E+06 3.5055003E+06 3.5055003E+06 3.4982404E+06 3.4912139E+06 3.4982404E+06 3.4912139E+06 3.4982404E+06 3.4982404E+06 3.4912139E+06 3.4848951E+06 3.4801119E+06 3.4768060E+06	Enthalpy (Btu/lb) 604.58 604.63 604.69 604.71 604.73 604.73 604.73 604.71 604.70 604.69 604.65 604.65 604.65 604.65 604.65 604.65 604.65 604.83 604.77 604.80 604.83 604.79 604.63 604.52 604.52 604.57 604.63 604.72 604.75
0.61010	5.7426034E+03	3.4729615E+06	604.77
0.62004	5.7325199E+03	3.4671373E+06	604.82
0.63014	5.7211947E+03	3.4605989E+06	604.87
0.64006	5.7162332E+03	3.4576773E+06	604.89
0.65003	5.7176191E+03	3.4583568E+06	604.86
0.66012	5.7204578E+03	3.4598599E+06	604.82
0.67003	5.7223865E+03	3.4608454E+06	604.79
0.68006	5.7257890E+03	3.4626698E+06	604.75

TABLE 6.2.1-28A (SHEET 6 OF 7)

Time <u>(s)</u>	Mass Flow _(lb/s)_	Energy Flow (Btu/s)	Average Enthalpy (Btu/lb)
	5.7317272E+03 5.7366181E+03 5.7366207E+03 5.7366207E+03 5.7320245E+03 5.7251012E+03 5.7172129E+03 5.7094890E+03 5.6997968E+03 5.6960407E+03 5.6960407E+03 5.6845992E+03 5.6845992E+03 5.6845992E+03 5.6942403E+03 5.6942403E+03 5.6970829E+03 5.6929918E+03 5.6929918E+03 5.6929918E+03 5.6929918E+03 5.6929918E+03 5.6929918E+03 5.6929918E+03 5.6929918E+03 5.6929918E+03 5.6708125E+03 5.6735480E+03 5.6735480E+03 5.6715705E+03 5.6735480E+03 5.6715705E+03 5.67422227E+03 5.6735480E+03 5.6745795E+03 5.67427925E+03 5.6542380E+03 5.6578617E+03 5.65426635E+03 5.6525971E+03 5.6325971E+03	(Btu/s) 3.4659330E+06 3.4686004E+06 3.4684965E+06 3.4657852E+06 3.4571775E+06 3.4526916E+06 3.4491984E+06 3.4449350E+06 3.4447350E+06 3.4386253E+06 3.4379041E+06 3.4399046E+06 3.4454743E+06 3.4454743E+06 3.4454743E+06 3.445899E+06 3.4491984E+06 3.445889E+06 3.429216E+06 3.4396914E+06 3.4396914E+06 3.4396914E+06 3.4393170E+06 3.4293170E+06 3.4293170E+06 3.4293170E+06 3.4293052E+06 3.4293052E+06 3.4293052E+06 3.4287262E+06 3.4287262E+06 3.4287262E+06 3.4281936E+06 3.4286424E+06 3.4181930E+06 3.4197733E+06 3.4107498E+06 3.4107498E+06 3.4107498E+06 3.4107498E+06 3.4990528E+06 3.4107498E+06 3.4107498E+06 3.4107498E+06 3.4107498E+06 3.43990528E+06	Enthalpy (Btu/lb) 604.69 604.64 604.62 604.64 604.70 604.75 604.75 604.75 604.77 604.79 604.78 604.67 604.63 604.61 604.61 604.62 604.63 604.63 604.68 604.68 604.68 604.68 604.55 604.55 604.55 604.53 604.55 604.53 604.53 604.53 604.53 604.53 604.53 604.53 604.53 604.53 604.53 604.53 604.53 604.53 604.53 604.53
1.40023 1.45004 1.50006 1.55004 1.60010	5.6056208E+03 5.5990362E+03 5.5888967E+03 5.5798754E+03 5.5703163E+03	3.3874824E+06 3.3833512E+06 3.3772132E+06 3.3717233E+06 3.3659471E+06	604.30 604.27 604.27 604.26 604.26

TABLE 6.2.1-28A (SHEET 7 OF 7)

Time (s)	Mass Flow (lb/s)	Energy Flow (Btu/s)	Average Enthalpy <u>(Btu/lb)</u>	
1.65004	5.5670531E+03	3.3637635E+06	604.23	
1.70008	5.5583218E+03	3.3584925E+06	604.23	
1.75012	5.5601180E+03	3.3591994E+06	604.16	
1.80010	5.5588433E+03	3.3581573E+06	604.11	
1.85006	5.5580249E+03	3.3573797E+06	604.06	
1.90001	5.5513019E+03	3.3532687E+06	604.05	
1.95012	5.5450010E+03	3.3494269E+06	604.04	
2.00000	5.5358334E+03	3.3439853E+06	604.06	

TABLE 6.2.1-29 (SHEET 1 OF 4)

BLOWDOWN MASS AND ENERGY RELEASES DOUBLE-ENDED PUMP SUCTION

		PATH NO. 1		PATH NO. 2
 -	`	of DEPS break)	`	e of DEPS Break)
Time	Flowrate	Energy	Flowrate	Energy
<u>(s)</u>	<u>lbm/s</u>	thousands Btu/s	<u>lbm/s</u>	thousands Btu/s
0.000	0.000	0.000	0.000	0.000
0.001	3946.600	2213.384	3847.100	2155.872
0.002	8319.000	4667.548	6074.500	3379.177
0.003	12253.500	6874.495	8559.000	4760.112
0.004	16370.200	9179.306	11258.100	6261.165
0.005	20028.700	11221.876	13620.500	7574.885
0.006	23257.200	13018.687	15753.200	8760.792
0.007	26436.000	14781.365	17975.000	9996.153
0.101	38289.600	21348.758	18542.300	10324.892
0.201	42923.600	24074.480	24115.500	13409.725
0.301	44108.800	24947.622	24725.900	13760.861
0.402	44790.700	25611.528	24193.600	13479.868
0.501	42452.100	24571.020	23257.000	12969.066
0.601	41819.700	24491.988	22410.700	12503.992
0.702	42149.900	24953.210	21648.800	12082.284
0.802	42044.700	25131.474	21058.000	11755.334
0.902	41576.400	25070.800	20631.600	11520.109
1.002	40989.400	24926.252	20335.000	11355.887
1.101	40354.200	24742.630	20087.700	11219.069
1.201	39639.500	24508.082	19981.100	11160.212
1.301	38880.700	24248.766	19927.800	11131.027
1.401	38126.600	23999.804	19899.200	11115.208
1.502	37226.500	23667.804	19868.200	11097.628
1.601	35972.300	23113.632	19846.200	11084.905
1.701	34766.400	22586.332	19839.200	11080.565
1.802	33507.100	22012.388	19833.300	11076.968
1.902	32367.500	21491.938	19805.200	11060.887
2.002	31249.800	20955.764	19727.800	11017.105
2.101	30202.400	20434.468	19623.100	10958.258
2.201	29228.400	19934.532	19503.500	10891.566
2.301	28306.800	19447.092	19348.800	10805.427
2.401	27398.100	18950.844	18935.200	10573.716
2.501	26455.600	18417.160	18639.000	10409.349

TABLE 6.2.1-29 (SHEET 2 of 4)

	BREAK P	ATH NO. 1	BREAK F	PATH NO. 2
	(from SG side	of DEPS break)	(from RCP side	e of DEPS Break)
Time	Flowrate	Energy	Flowrate	Energy
<u>(s)</u>	<u>lbm/s</u>	thousands Btu/s	<u>lbm/s</u>	thousands Btu/s
2.601	25393.400	17788.690	18442.800	10301.389
2.701	24357.200	17173.500	18232.800	10185.621
2.801	23417.200	16617.960	17986.900	10049.795
2.901	22544.500	16095.738	17732.600	9909.640
3.001	21745.800	15609.267	17490.600	9776.739
3.102	21041.300	15172.657	17236.300	9637.167
3.201	20419.100	14777.543	17024.500	9521.626
3.301	19863.600	14415.904	16806.400	9402.622
3.401	19292.700	14029.571	16600.400	9290.511
3.501	18663.700	13592.869	16403.200	9183.516
3.601	18160.800	13241.620	16209.000	9078.271
3.702	17805.700	12987.854	16031.600	8982.589
3.801	17489.800	12752.802	15863.500	8892.147
3.902	17216.300	12543.659	15688.700	8797.898
4.001	17021.800	12387.140	15527.600	8711.463
4.201	16710.200	12115.103	15232.400	8553.681
4.400	16515.800	11915.966	14973.000	8416.208
4.600	16382.800	11752.949	14723.100	8283.999
4.800	16339.200	11646.704	14512.400	8173.967
5.000	16393.200	11602.067	14440.700	8145.812
5.201	16458.700	11562.862	15618.600	8818.020
5.400	16575.300	11558.639	15527.600	8776.119
5.600	16740.100	11589.474	15415.900	8721.972
5.800	16895.000	11623.031	15293.000	8660.437
6.000	17015.400	11648.306	15104.200	8561.191
6.200	17094.600	11660.240	14884.400	8443.604
6.401	19098.600	13074.699	14695.600	8342.916
6.600	15337.500	11382.634	14567.600	8274.761
6.801	14813.900	11156.005	14282.600	8114.348
7.000	15048.300	11210.722	14145.400	8038.984
7.200	15361.700	11276.829	13954.300	7931.919
7.400	15748.300	11370.368	13783.400	7834.016
7.601	16161.200	11487.324	13559.700	7704.940
7.801	16477.400	11554.377	13347.800	7584.514
8.000	16782.600	11624.411	13197.200	7499.874
8.200	17064.400	11694.815	12982.600	7377.614

TABLE 6.2.1-29 (SHEET 3 of 4)

		ATH NO. 1 of DEPS break)		PATH NO. 2 e of DEPS Break)
Time	Flowrate	Energy	Flowrate	Energy
	Ibm/s	thousands Btu/s	Ibm/s	thousands Btu/s
<u>(s)</u>	10111/5	tilousarius btu/s	<u>10111/5</u>	tilousarius blu/s
8.400	17361.100	11786.569	12761.100	7251.813
8.600	17735.100	11939.677	12586.700	7153.691
8.801	18159.700	12123.226	12344.400	7016.289
9.000	18155.300	12066.221	12133.600	6897.901
9.200	17672.100	11737.936	11958.500	6799.769
9.401	16721.000	11115.496	11753.400	6682.800
9.601	15405.500	10411.341	11712.200	6660.538
9.800	14365.600	9946.658	11500.600	6535.169
10.001	13987.100	9776.742	11313.000	6426.781
10.201	13596.200	9562.549	11247.200	6392.091
10.400	13262.200	9394.736	11005.200	6251.513
10.601	13064.600	9287.898	10861.100	6169.429
10.801	12782.600	9123.655	10701.600	6078.292
11.000	12321.000	8845.054	10495.200	5958.064
11.201	11849.400	8557.324	10370.900	5885.762
11.400	11141.300	8089.585	10178.000	5771.663
11.601	9795.600	7216.975	10123.700	5737.092
11.801	8968.100	6743.041	10046.200	5682.767
12.000	8507.600	6468.497	9909.800	5593.485
12.201	7889.400	6152.172	9972.100	5619.842
12.401	7641.400	6058.317	9748.400	5474.546
12.601	7493.600	5970.287	9762.900	5471.703
12.800	7209.400	5792.173	9572.900	5335.134
13.001	6948.600	5634.834	9517.800	5255.479
13.201	6626.600	5426.060	9512.700	5176.978
13.400	6278.000	5210.075	9266.900	4952.279
13.601	6006.900	5027.989	9513.300	4986.285
13.800	5725.100	4832.479	9314.900	4804.915
14.001	5500.100	4671.335	8926.200	4527.542
14.200	5259.700	4494.066	9301.100	4646.362
14.400	4976.700	4283.420	8875.400	4393.291
14.602	4743.800	4106.409	7462.000	3634.623
14.801	4599.200	3968.906	9176.000	4357.596
15.000	4460.200	3827.137	13141.700	6343.803
15.200	4246.300	3663.303	11795.400	5761.067
15.401	4117.300	3614.793	7675.300	3755.504
15.600	4223.700	3679.953	4656.200	2239.710

TABLE 6.2.1-29 (SHEET 4 of 4)

		PATH NO. 1		PATH NO. 2
	`	of DEPS break)	•	e of DEPS Break)
Time	Flowrate	Energy	Flowrate	Energy
<u>(s)</u>	<u>lbm/s</u>	thousands Btu/s	<u>lbm/s</u>	thousands Btu/s
15.800	3964.200	3613.322	9864.700	4327.320
16.000	3332.500	3391.530	9541.400	4265.034
16.200	2913.800	3316.567	7352.900	3324.880
16.400	2610.800	3195.538	5615.700	2551.255
16.600	2309.800	2863.751	4684.600	2082.656
16.801	2080.200	2590.111	6960.200	2829.077
17.000	1880.800	2348.553	7062.000	2798.790
17.201	1720.500	2153.588	7114.500	2782.187
17.401	1583.100	1985.670	6358.200	2461.655
17.601	1458.400	1832.525	5725.900	2182.913
17.801	1343.000	1690.322	5336.600	1990.871
18.001	1238.200	1560.737	5087.200	1844.413
18.200	1222.000	1541.643	4839.400	1697.595
18.401	1094.500	1385.090	4588.300	1553.727
18.600	1021.500	1294.326	4307.200	1408.681
18.800	979.600	1241.900	4034.300	1276.978
19.000	907.000	1150.252	3761.300	1155.645
19.201	789.500	1001.381	3452.900	1033.423
19.401	632.600	803.720	3061.900	896.668
19.600	656.500	834.921	2687.300	773.064
19.801	663.000	843.023	2262.700	641.353
20.001	563.600	716.552	1623.300	455.481
20.200	494.300	628.878	1149.000	320.160
20.401	446.500	568.307	654.300	181.636
20.601	391.000	498.141	281.100	77.972
20.800	346.800	442.014	15.400	4.273
21.001	309.100	394.141	103.000	28.788
21.200	284.100	362.388	82.700	23.272
21.400	272.400	347.662	275.200	77.099
21.600	240.100	306.527	267.800	74.073
21.801	207.200	264.697	0.000	0.000
22.001	167.500	214.144	0.000	0.000
22.200	134.700	172.321	0.000	0.000
22.401	123.200	157.731	0.000	0.000
22.600	101.800	130.473	0.000	0.000
22.800	47.200	60.581	0.000	0.000
23.001	0.000	0.000	0.000	0.000

TABLE 6.2.1-30 (SHEET 1 OF 4)

BLOWDOWN MASS AND ENERGY RELEASES DOUBLE-ENDED HOT LEG GUILLOTINE

	BREAK PATH NO. 1		BREAK PATH NO. 2		
	(from vessel sid	de of DEHL break)	(from SG side	e of DEHL Break)	
Time	Flowrate	Energy	Flowrate	Energy	
<u>(s)</u>	<u>lbm/s</u>	thousands Btu/s	<u>lbm/s</u>	thousands Btu/s	
0.000	0.00	0.00	0.00	0.00	
0.000	0.00	0.00	0.00	0.00	
0.001	2171.70	1413.90	2171.60	1413.85	
0.002	4537.50	2960.63	4412.60	2864.48	
0.101	42896.90	28286.88	26992.90	17487.46	
0.201	36153.90	24036.17	24380.00	15694.78	
0.302	36261.80	24123.29	21571.80	13699.80	
0.402	35379.60	23551.49	20233.40	12628.28	
0.501	34977.20	23304.98	19454.10	11937.60	
0.602	34564.30	23065.72	18948.50	11454.52	
0.702	34173.90	22853.01	18556.10	11080.26	
0.802	33658.70	22568.82	18280.40	10805.63	
0.902	32996.20	22195.30	18067.30	10589.63	
1.002	32318.90	21817.01	17919.50	10428.70	
1.101	31738.40	21506.43	17865.00	10334.16	
1.202	31350.40	21333.05	17847.80	10269.68	
1.302	30918.80	21122.79	17888.60	10245.45	
1.402	30393.70	20823.86	17959.30	10244.94	
1.502	29865.90	20504.41	18028.30	10248.92	
1.601	29419.50	20227.70	18108.30	10264.80	
1.701	29077.90	20022.70	18185.40	10283.90	
1.801	28728.90	19814.29	18250.90	10301.16	
1.902	28258.50	19517.49	18295.70	10311.01	
2.002	27720.60	19166.19	18316.90	10311.65	
2.101	27214.80	18833.46	18321.70	10306.36	
2.202	26789.00	18556.69	18311.70	10295.44	
2.302	26422.90	18321.22	18286.30	10278.09	
2.401	26043.60	18072.76	18242.60	10252.32	
2.502	25617.60	17784.15	18183.10	10219.09	
2.602	25189.20	17486.03	18108.60	10178.60	
2.702	24822.00	17226.87	18024.80	10133.61	
2.802	24511.00	17004.27	17934.80	10085.75	
2.902	24215.50	16788.35	17837.90	10034.35	
3.001	23936.30	16578.70	17733.10	9978.79	

TABLE 6.2.1-30 (SHEET 2 OF 4)

		PATH NO. 1		PATH NO. 2
-	•	de of DEHL break)	•	e of DEHL Break)
Time	Flowrate	Energy	Flowrate	Energy
<u>(s)</u>	<u>lbm/s</u>	thousands Btu/s	<u>lbm/s</u>	thousands Btu/s
3.102	23659.80	16365.28	17622.50	9920.02
3.201	23396.00	16155.66	17508.10	9859.14
3.302	23179.20	15974.69	17389.60	9795.99
3.403	22992.20	15811.25	17270.30	9732.31
3.502	22821.50	15656.48	17149.20	9667.49
3.601	22682.30	15520.36	17026.30	9601.57
3.702	22550.20	15385.68	16899.70	9533.60
3.802	22437.70	15262.24	16773.50	9465.81
3.902	22352.50	15155.65	16648.50	9398.64
4.002	22285.70	15059.58	16523.30	9331.46
4.201	22237.50	14920.50	16274.40	9198.21
4.401	22343.10	14877.12	16026.00	9066.06
4.601	22468.90	14845.11	15774.40	8933.11
4.801	22649.60	14862.39	15514.10	8796.44
5.000	22904.60	14939.88	15304.40	8690.74
5.201	23247.60	15075.02	15075.10	8573.81
5.401	13149.70	9672.27	14707.90	8378.06
5.601	17298.40	12349.15	14462.20	8254.32
5.800	17527.30	12373.66	14183.00	8112.51
6.000	17758.00	12456.65	13913.90	7976.93
6.200	18022.60	12480.00	13620.60	7826.23
6.400	18272.60	12552.55	13354.00	7690.85
6.600	18583.60	12605.83	13076.60	7548.18
6.800	18879.60	12660.19	12786.60	7397.68
7.001	19155.30	12743.80	12524.80	7263.83
7.201	19484.00	12825.86	12252.70	7122.76
7.400	19865.70	12971.89	11980.60	6980.68
7.601	20190.10	13082.44	11719.30	6844.64
7.801	22377.20	14339.24	11465.90	6712.11
8.000	30164.60	19269.67	11211.50	6578.10
8.200	28343.10	17972.04	10949.80	6440.00
8.401	27709.40	17452.65	10648.40	6277.73
8.601	27220.50	17098.87	10288.00	6083.40
8.801	26662.50	16749.49	9983.60	5927.46
9.001	26201.00	16484.27	9625.40	5738.55
9.202	24370.30	15393.04	9278.60	5558.30

TABLE 6.2.1-30 (SHEET 3 OF 4)

	BREAK I	PATH NO. 1	BREAK	PATH NO. 2
	(from vessel sig	de of DEHL break)	(from SG side	e of DEHL Break)
Time	Flowrate	Energy	Flowrate	Energy
<u>(s)</u>	<u>lbm/s</u>	thousands Btu/s	<u>lbm/s</u>	thousands Btu/s
9.400	24022.20	15231.08	8925.00	5374.67
9.602	23754.00	15058.04	8582.70	5200.08
9.801	23506.50	14886.21	8243.90	5027.26
10.002	23173.20	14660.18	7920.00	4864.64
10.201	15252.20	9446.14	7572.30	4686.63
10.401	16136.80	9820.34	7304.80	4556.20
10.601	9585.90	6879.72	7060.40	4441.43
10.800	10817.90	7687.76	6923.40	4375.90
11.001	10496.20	7546.23	6858.40	4346.84
11.200	10651.30	7606.32	6760.10	4286.48
11.400	10695.20	7626.37	6711.10	4260.75
11.602	10639.80	7610.49	6601.40	4196.91
11.801	10518.20	7574.66	6508.50	4144.15
12.001	10304.30	7495.80	6384.60	4075.23
12.200	10012.60	7368.45	6223.60	3983.05
12.402	9685.20	7213.39	6035.50	3882.87
12.601	9317.50	7029.79	5762.40	3733.41
12.801	8889.70	6805.87	5466.20	3582.76
13.000	8467.90	6566.26	5119.30	3399.86
13.201	8040.30	6299.14	4775.30	3223.65
13.401	7685.00	6040.87	4450.30	3052.80
13.602	7339.90	5743.69	4163.50	2898.72
13.800	7105.40	5489.41	3920.30	2764.99
14.001	6901.00	5256.23	3709.80	2645.07
14.200	6728.80	5050.93	3525.60	2538.46
14.401	6595.80	4908.84	3349.10	2441.24
14.601	5244.90	4761.57	3204.60	2362.88
14.800	4280.60	4531.14	3084.70	2294.16
15.000	3819.80	4252.30	2996.00	2232.91
15.200	3535.70	3948.31	2914.90	2174.74
15.400	3302.40	3668.29	2815.50	2117.82
15.601	3092.60	3457.57	2690.50	2058.94
15.800	2875.40	3276.10	2548.20	1997.75
16.000	2680.70	3124.02	2392.10	1934.47
16.201	2481.60	2978.91	2241.50	1875.60
16.401	2395.80	2907.75	2099.80	1820.19
16.600	2318.00	2797.22	1963.40	1761.61

TABLE 6.2.1-30 (SHEET 4 OF 4)

	BREAK I	PATH NO. 1	BREAK	PATH NO. 2
	(from vessel sid	de of DEHL break)	(from SG side	e of DEHL Break)
Time	Flowrate	Energy	Flowrate	Energy
<u>(s)</u>	<u>lbm/s</u>	thousands Btu/s	<u>lbm/s</u>	thousands Btu/s
16.800	2229.30	2657.72	1816.90	1711.56
17.000	2182.40	2549.03	1653.20	1660.58
17.200	2116.90	2443.35	1493.90	1612.33
17.401	2022.00	2341.32	1357.60	1547.46
17.601	1898.60	2235.50	1245.20	1470.71
17.800	1782.60	2150.31	1152.20	1393.91
18.000	1690.80	2054.20	1165.20	1423.48
18.201	1637.90	1975.67	1034.20	1273.66
18.400	1577.60	1914.02	972.40	1201.47
18.601	1486.00	1818.85	922.40	1142.39
18.800	1413.60	1739.58	870.10	1079.63
19.000	1340.60	1656.26	823.30	1023.25
19.200	1287.00	1591.26	782.10	973.34
19.400	1248.00	1538.83	741.60	923.95
19.600	1205.00	1488.46	706.90	881.80
19.800	1156.10	1433.16	681.30	850.65
20.000	1097.50	1365.85	653.80	817.09
20.201	1056.40	1312.43	629.80	787.74
20.400	1166.60	1443.86	608.00	761.07
20.600	1114.80	1376.40	586.00	734.15
20.800	1060.30	1309.00	565.40	708.93
21.001	1004.10	1241.06	547.00	686.23
21.200	933.20	1159.38	527.40	662.01
21.400	847.40	1059.64	506.10	635.81
21.600	788.70	980.47	481.80	605.72
21.800	708.30	887.43	459.30	577.85
22.001	626.70	787.49	437.20	550.32
22.201	553.60	693.34	436.00	549.28
22.401	457.80	574.51	433.70	546.58
22.601	346.20	434.65	434.00	547.13
22.801	271.60	342.18	431.70	544.38
23.000	170.50	217.34	412.50	520.26
23.200	63.10	82.11	367.10	463.24
23.400	0.00	0.00	286.50	362.00
23.600	0.00	0.00	203.10	257.51
23.800	0.00	0.00	52.10	66.58
24.001	0.00	0.00	0.00	0.00

TABLE 6.2.1-31 (SHEET 1 OF 5)

REFLOOD MASS AND ENERGY RELEASES DOUBLE-ENDED PUMP SUCTION - MINIMUM SAFETY INJECTION

	BREAK PA	ATH NO. 1 of DEPS break)	BREAK PA	ATH NO. 2 of DEPS Break)
Time	Flowrate	Energy thousands	Flowrate	Energy thousands
<u>(s)</u>	<u>lbm/s</u>	<u>Btu/s</u>	<u>lbm/s</u>	Btu/s
23.001	0.000	0.000	0.000	0.000
23.476	0.000	0.001	0.000	0.000
23.676	0.000	0.001	0.000	0.000
23.776	0.000	0.001	0.000	0.000
23.876	0.000	0.001	0.000	0.000
23.976	0.000	0.001	0.000	0.000
24.026	0.000	0.001	0.000	0.000
24.126	63.700	75.121	0.000	0.000
24.226	32.300	38.060	0.000	0.000
24.351	29.200	34.476	0.000	0.000
24.451	35.800	42.239	0.000	0.000
24.551	43.200	51.006	0.000	0.000
24.651	48.700	57.402	0.000	0.000
24.751	53.700	63.367	0.000	0.000
24.851	58.500	68.965	0.000	0.000
24.951	62.900	74.252	0.000	0.000
25.051	67.200	79.292	0.000	0.000
25.151	71.300	84.112	0.000	0.000
25.176	72.300	85.285	0.000	0.000
25.251	75.200	88.737	0.000	0.000
25.351	79.000	93.190	0.000	0.000
25.451	82.600	97.489	0.000	0.000
25.551	86.200	101.647	0.000	0.000
25.651	89.600	105.679	0.000	0.000
25.751	92.900	109.593	0.000	0.000
25.851	96.100	113.400	0.000	0.000
25.951	99.200	117.107	0.000	0.000
26.051	102.300	120.722	0.000	0.000
27.051	129.500	152.886	0.000	0.000
28.051	152.400	179.953	0.000	0.000
29.051	470.400	558.117	4630.300	652.018
29.351	510.200	605.803	4988.800	725.530
30.051	518.600	615.927	5049.900	747.599

TABLE 6.2.1-31 (SHEET 2 of 5)

	BREAK P	ATH NO. 1	BREAK PA	ATH NO. 2
	(from SG side	of DEPS break)	(from RCP side	of DEPS Break)
Time	Flowrate	Energy	Flowrate	Energy
	. ,	thousands	,	thousands
<u>(s)</u>	<u>lbm/s</u>	<u>Btu/s</u>	<u>lbm/s</u>	<u>Btu/s</u>
31.051	511.900	607.996	4992.100	742.445
32.051	503.800	598.224	4919.200	734.449
33.051	495.300	588.044	4842.000	725.623
33.351	492.700	584.979	4818.600	722.903
34.051	486.800	577.862	4763.900	716.504
35.051	478.400	567.860	4686.300	707.350
36.051	470.300	558.127	4610.000	698.296
37.051	462.400	548.702	4535.600	689.410
38.051	454.800	539.599	4463.000	680.730
38.651	450.400	534.292	4420.500	675.629
39.051	447.500	530.819	4392.500	672.274
40.101	307.000	363.294	732.600	225.778
41.101	308.000	364.431	710.600	221.276
42.101	321.800	380.854	659.500	213.846
43.101	332.100	393.150	615.700	207.870
44.101	198.000	233.879	1364.300	320.259
45.101	196.600	232.271	1345.900	317.424
46.001	475.100	563.610	336.400	271.884
46.101	494.600	587.062	345.200	284.180
47.101	529.800	629.367	360.700	306.629
48.101	513.400	609.746	353.100	296.397
49.101	495.800	588.669	345.000	285.429
50.101	479.800	569.533	337.600	275.502
51.101	464.600	551.338	330.600	266.097
51.801	454.400	539.087	325.800	259.785
52.101	450.100	533.949	323.800	257.144
53.101	436.200	517.359	317.400	248.637
54.101	422.900	501.517	311.300	240.543
55.101	410.200	486.339	305.400	232.815
56.101	398.100	471.788	299.800	225.433
57.101	386.400	457.832	294.400	218.377
58.101	375.100	444.443	289.300	211.632
59.101	364.400	431.595	284.300	205.182
60.101	354.000	419.262	279.600	199.013
61.101	344.100	407.418	275.000	193.112

TABLE 6.2.1-31 (SHEET 3 of 5)

	BREAK PA	ATH NO. 1	BREAK P	ATH NO. 2
	(from SG side of	of DEPS break)	(from RCP side	of DEPS Break)
Time	Flowrate	Energy	Flowrate	Energy
	,	thousands	,	thousands
<u>(s)</u>	<u>lbm/s</u>	Btu/s	<u>lbm/s</u>	<u>Btu/s</u>
62.101	334.500	396.057	270.700	187.473
63.101	325.400	385.150	266.500	182.079
64.101	316.600	374.679	262.500	176.918
65.101	308.100	364.627	258.700	171.982
66.101	300.000	354.979	255.000	167.262
67.101	292.200	345.721	251.500	162.749
67.301	290.700	343.915	250.800	161.870
68.101	284.800	336.839	248.100	158.435
69.101	277.600	328.320	244.900	154.312
70.101	270.700	320.151	241.800	150.373
71.101	264.100	312.321	238.900	146.612
72.101	257.800	304.817	236.100	143.020
73.101	251.700	297.631	233.400	139.593
74.101	245.900	290.750	230.800	136.324
75.101	240.400	284.165	228.300	133.208
76.101	235.100	277.867	226.000	130.237
77.101	230.000	271.845	223.800	127.408
78.101	225.200	266.090	221.600	124.713
79.101	220.500	260.605	219.600	122.155
80.101	216.100	255.371	217.700	119.723
81.101	211.900	250.368	215.800	117.407
82.101	207.900	245.595	214.100	115.205
83.101	204.000	241.045	212.400	113.114
84.101	200.400	236.712	210.800	111.129
86.101	193.600	228.666	207.900	107.461
88.101	187.400	221.399	205.300	104.169
89.701	183.000	216.107	203.400	101.785
90.101	181.900	214.854	202.900	101.221
92.101	176.900	208.978	200.800	98.590
94.101	172.500	203.719	198.900	96.247
96.101	168.500	199.028	197.300	94.166
98.101	165.000	194.857	195.800	92.322
100.101	161.900	191.163	194.500	90.695
102.101	159.100	187.882	193.300	89.254
104.101	156.700	184.992	192.300	87.987

TABLE 6.2.1-31 (SHEET 4 of 5)

	BREAK PA	ATH NO. 1	BREAK PA	ATH NO. 2
	(from SG side of	of DEPS break)	(from RCP side	of DEPS Break)
Time	Flowrate	Energy thousands	Flowrate	Energy thousands
<u>(s)</u>	<u>lbm/s</u>	<u>Btu/s</u>	<u>lbm/s</u>	<u>Btu/s</u>
106.101	154.500	182.458	191.400	86.877
108.101	152.700	180.248	190.600	85.909
110.101	151.000	178.328	190.000	85.067
112.101	149.600	176.669	189.400	84.337
114.101	148.400	175.245	188.900	83.708
116.101	147.400	174.031	188.400	83.168
118.101	146.500	173.004	188.100	82.708
118.901	146.200	172.642	187.900	82.545
120.101	145.800	172.145	187.700	82.319
122.101	145.200	171.436	187.500	81.993
124.101	144.700	170.859	187.300	81.722
126.101	144.300	170.400	187.100	81.500
128.101	144.000	170.046	186.900	81.322
130.101	143.800	169.784	186.800	81.183
132.101	143.700	169.606	186.700	81.077
134.101	143.600	169.500	186.700	81.002
136.101	143.500	169.458	186.600	80.953
138.101	143.600	169.473	186.600	80.928
140.101	143.600	169.539	186.600	80.923
142.101	143.700	169.648	186.600	80.936
144.101	143.800	169.797	186.700	80.965
146.101	144.000	169.979	186.700	81.008
148.101	144.200	170.191	186.700	81.064
150.101	144.400	170.429	186.800	81.130
151.201	144.500	170.561	186.800	81.167
152.101	144.600	170.670	186.800	81.198
154.101	144.800	170.927	186.900	81.273
156.101	145.000	171.202	187.000	81.355
158.101	145.300	171.493	187.000	81.444
160.101	145.500	171.797	187.100	81.539
162.101	145.800	172.114	187.200	81.638
164.101	146.100	172.440	187.300	81.742
166.101	146.300	172.775	187.400	81.850
168.101	146.600	173.118	187.500	81.960
170.101	146.900	173.466	187.500	82.074

TABLE 6.2.1-31 (SHEET 5 OF 5)

	BREAK PATH NO. 1		BREAK PATH NO. 2	
	(from SG side	of DEPS break)	(from RCP side	of DEPS Break)
Time	Flowrate	Energy	Flowrate	Energy
<u>(s)</u>	<u>lbm/s</u>	thousands <u>Btu/s</u>	<u>lbm/s</u>	thousands <u>Btu/s</u>
(3)	10111/3	<u> </u>	10111/3	<u>Dtu/5</u>
172.101	147.200	173.820	187.600	82.190
174.101	147.500	174.178	187.700	82.307
176.101	147.800	174.539	187.800	82.427
178.101	148.100	174.904	187.900	82.547
180.101	148.500	175.270	188.000	82.669
182.101	148.800	175.639	188.100	82.792
184.101	149.100	176.008	188.200	82.916
184.901	149.200	176.157	188.300	82.966
186.101	149.400	176.379	188.300	83.040
188.101	149.700	176.751	188.400	83.165
190.101	150.000	177.123	188.500	83.291
192.101	150.300	177.496	188.600	83.417
194.101	150.700	177.869	188.700	83.544
196.101	151.000	178.242	188.800	83.671
198.101	151.300	178.616	188.900	83.798
200.101	151.600	178.989	189.000	83.926
202.101	151.900	179.342	189.100	84.046
204.101	152.200	179.692	189.200	84.166
206.101	152.500	180.044	189.300	84.288
208.101	152.800	180.399	189.400	84.411
210.101	153.100	180.757	189.500	84.535
212.101	153.400	181.117	189.600	84.660
214.101	153.700	181.481	189.700	84.788
216.101	154.000	181.848	189.800	84.917
218.101	154.300	182.220	189.900	85.048
219.901	154.600	182.558	190.000	85.167

TABLE 6.2.1-32

REFLOOD MASS AND ENERGY RELEASES DOUBLE-ENDED PUMP SUCTION - MAXIMUM SAFETY INJECTION

THIS TABLE HAS BEEN DELETED.

TABLE 6.2.1-33

PRINCIPAL PARAMETERS DURING REFLOOD DOUBLE-ENDED PUMP SUCTION - MINIMUM SAFETY INJECTION

	Enthalpy <u>Btu/lbm</u>	0.00	89.73	89.73	89.73	89.73	89.73	89.73	89.73	89.73	89.73	89.73	92.16	97.99	97.99	97.99	97.99	97.99	97.99	97.99	97.99	97.99	97.99	97.99
Spill		0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Injection Accum	(s/mql)	0.00	8087.40	8039.50	7735.10	7553.60	6349.80	6063.50	5786.30	5264.60	1297.90	1207.60	1240.20	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Total		0.00	8087.40	8039.50	7735.10	7553.60	6349.80	6063.50	5786.30	5264.60	1297.90	1207.60	1756.60	465.50	458.60	444.00	463.30	482.80	496.10	505.90	510.80	513.50	514.50	514.80
	Flow Fraction	0.25	0.00	0.00	0.32	0.35	09.0	09.0	0.59	0.58	0.51	0.53	0.42	0.62	0.62	0.62	0.61	09.0	0.59	0.57	0.55	0.53	0.52	0.52
	Downcomer <u>Height (ft)</u>	00.00	1.62	1.53	4.97	7.81	16.11	16.12	16.12	16.12	16.12	16.12	16.12	16.01	15.98	15.58	13.95	12.12	10.81	9.88	9.55	9.64	10.01	10.57
	Core Height (ft)	0.00	99.0	1.06	1.50	1.63	2.01	2.24	2.50	3.01	3.12	3.33	3.45	3.50	3.51	3.60	4.00	4.52	2.00	5.53	00.9	6.52	7.00	7.54
	Carryover <u>Fraction</u>	0.00	0.00	0.00	0.30	0.42	0.63	0.68	0.71	0.73	0.73	0.73	0.73	0.74	0.74	0.74	0.74	0.74	0.74	0.73	0.73	0.73	0.73	0.73
ling	Rate <u>in./s</u>	0.00	22.81	25.30	2.70	2.61	4.99	4.64	4.32	3.90	3.04	3.17	2.41	4.00	4.10	4.25	3.67	3.00	2.46	1.99	1.69	1.50	1.42	1.39
Flooding	Temp (°F)	175.10	172.90	171.80	171.20	171.40	172.20	172.60	173.50	176.30	177.10	179.10	180.60	181.40	181.50	182.50	187.90	196.50	206.40	218.10	227.90	237.40	245.50	253.50
	Time (s)	23.00	23.80	24.00	25.20	26.10	29.40	31.10	33.40	38.70	40.10	43.10	45.10	46.00	46.10	47.10	51.80	59.10	67.30	78.10	89.70	104.10	118.90	136.10

TABLE 6.2.1-34

PRINCIPAL PARAMETERS DURING REFLOOD DOUBLE-ENDED PUMP SUCTION - MAXIMUM SAFETY INJECTION

THIS TABLE HAS BEEN DELETED.

TABLE 6.2.1-35 (SHEET 1 OF 2)

POST-REFLOOD MASS AND ENERGY RELEASES DOUBLE-ENDED PUMP SUCTION - MINIMUM SAFETY INJECTION

		ATH NO. 1		ATH NO. 2
	(from SG side of	of DEPS break)	(from RCP side	of DEPS Break)
Time	Flowrate	Energy	Flowrate	Energy
		thousands	,	thousands
<u>(s)</u>	<u>lbm/s</u>	<u>Btu/s</u>	<u>lbm/s</u>	<u>Btu/s</u>
220.0	235.200	293.278	285.700	137.415
225.0	234.900	292.950	286.000	137.277
230.0	234.700	292.584	286.300	137.146
235.0	234.300	292.178	286.600	137.024
240.0	234.000	291.732	286.900	136.910
245.0	234.500	292.446	286.400	136.546
250.0	234.100	291.895	286.800	136.453
255.0	233.600	291.301	287.300	136.370
260.0	233.100	290.661	287.800	136.297
265.0	233.500	291.134	287.400	135.982
270.0	232.900	290.378	288.000	135.932
275.0	232.200	289.571	288.700	135.893
280.0	232.500	289.840	288.500	135.621
285.0	231.700	288.904	289.200	135.609
290.0	231.800	289.018	289.100	135.370
295.0	230.900	287.947	290.000	135.386
300.0	230.900	287.904	290.000	135.180
305.0	230.800	287.755	290.100	134.996
310.0	230.600	287.498	290.300	134.835
315.0	230.300	287.131	290.600	134.698
320.0	229.900	286.652	291.000	134.584
325.0	229.400	286.056	291.500	134.495
330.0	229.600	286.332	291.300	134.217
335.0	228.900	285.453	292.000	134.188
340.0	228.900	285.404	292.000	133.979
345.0	228.700	285.162	292.200	133.812
350.0	228.300	284.719	292.600	133.687
355.0	227.800	284.058	293.100	133.609
360.0	227.800	284.071	293.100	133.386
365.0	227.600	283.779	293.300	133.227
370.0	227.100	283.171	293.800	133.136
375.0	226.300	282.212	294.600	133.121
380.0	226.600	282.517	294.300	132.832

TABLE 6.2.1-35 (SHEET 2 OF 2)

		ATH NO. 1		ATH NO. 2
Time a	•	of DEPS break)	`	of DEPS Break)
Time	Flowrate	Energy thousands	Flowrate	Energy thousands
<u>(s)</u>	<u>lbm/s</u>	Btu/s	<u>lbm/s</u>	Btu/s
385.0	225.700	281.480	295.200	132.833
390.0	225.700	281.456	295.200	132.614
395.0	225.100	280.694	295.800	132.554
400.0	225.000	280.526	295.900	132.365
405.0	224.700	280.117	296.300	132.237
410.0	224.200	279.581	296.700	132.135
415.0	224.200	279.595	296.700	131.914
420.0	223.800	279.028	297.100	131.818
425.0	223.500	278.726	297.400	131.664
430.0	223.100	278.136	297.800	131.572
435.0	222.900	277.940	298.000	131.394
620.4	222.900	277.940	298.000	131.394
620.5	96.600	119.717	424.300	162.981
625.0	96.500	119.561	424.500	162.781
1647.4	96.500	119.561	424.500	162.781
1647.5	76.500	87.981	444.400	60.003
2500.0	69.300	79.740	451.600	61.296
2500.1	69.300	79.739	369.700	53.269
3600.0	62.400	71.794	376.600	54.515
3600.1	50.700	58.358	388.300	38.048
3779.3	49.700	57.165	389.300	38.149
3779.4	49.700	57.164	389.300	38.149
4000.0	48.400	55.695	390.600	38.274
4000.1	49.300	56.776	389.700	45.991
10000.0	37.600	43.259	401.400	47.378
10000.1	37.400	43.051	401.600	45.391
100000.0	20.000	23.016	419.000	47.359
100000.1	19.600	22.581	419.400	39.019
1000000.0	8.400	9.676	430.600	40.062
1000000.1	8.400	9.631	430.600	37.917
10000000.0	2.600	3.016	436.400	38.423

TABLE 6.2.1-36

POST-REFLOOD MASS AND ENERGY RELEASES DOUBLE-ENDED PUMP SUCTION - MAXIMUM SAFETY INJECTION

THISTABLE HAS BEEN DELETED.

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TABLE 6.2.1-37

MASS BALANCE DOUBLE-ENDED PUMP SUCTION - MINIMUM SAFETY INJECTION

		Start of transient	End of	End of	End of	Broken loop SG	Intact loop SG	Final depressurization time
	Time (s)	0.0	23.00	23.00	219.90	620.48	1647.36	3600.00
		Mass (the	Mass (thousand lbm)					
Initial	In RCS and ACC	723.46	723.46	723.46	723.46	723.46	723.46	723.46
Added Mass	Pumped injection	0.00	00.00	0.00	89.65	298.27	833.18	1760.23
	Total added	0.00	00.00	00.00	89.65	298.27	833.18	1760.23
*** Total	*** Total Available ***	723.46	723.46	723.46	813.11	1021.73	1556.64	2483.69
Distribution	Reactor coolant	517.03	72.49	96.12	153.12	153.12	153.12	153.12
	Accumulator	206.43	135.52	111.89	0.00	00.00	0.00	0.00
	Total contents	723.46	208.01	208.01	153.12	153.12	153.12	153.12
Effluent	Break flow	0.00	515.44	515.44	648.47	857.09	1392.00	2319.06
	ECCS spill	0.00	00.00	0.00	0.00	00.00	0.00	0.00
	Total effluent	0.00	515.44	515.44	648.47	857.09	1392.00	2319.06
*** Total A	*** Total Accountable ***	723.46	723.45	723.45	801.59	1010.21	1545.12	2472.18

TABLE 6.2.1-38

MASS BALANCE DOUBLE-ENDED PUMP SUCTION - MAXIMUM SAFETY INJECTION

THIS TABLE HAS BEEN DELETED.

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TABLE 6.2.1-39

MASS BALANCE DOUBLE-ENDED HOT LEG GUILLOTINE

i		Start of transient	End of blowdown	End of refill
Time (s)	(8)	0.00	24.00	24.00
		Mass	Mass (thousand lbm)	'n
Initial	In RCS and ACC	723.46	723.46	723.46
Added Mass	Pumped Injection	0.00	0.00	0.00
	Total Added	0.00	0.00	0.00
Total Available	ilable***	723.46	723.46	723.46
Distribution	Reactor Coolant	517.03	109.45	132.34
	Accumulator	206.43	120.27	97.37
	Total Contents	723.46	229.71	229.71
Effluent	Break Flow	0.00	493.74	493.74
	ECCS Spill	0.00	0.00	0.00
	Total Effluent	0.00	493.74	493.74
Total Accountable	untable***	723.46	723.46	723.46

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TABLE 6.2.1-40

ENERGY BALANCE DOUBLE-ENDED PUMP SUCTION - MINIMUM SAFETY INJECTION

Intact loop Final SG depressurization equilibration time 1647.36 3600.00	877.93 877.93	81.64 172.49	146.28 261.51	15.69 15.69	243.62 449.69	1121.55 1327.62	40.55 40.55	0.00 0.00	4.13 3.33	61.2 46.9	36.11 26.95	131.73 97.61	273.72 215.33	854.41 1120.26	0.00 0.00	854.41 1120.26	1128.14 1335.59
Broken loop Int SG equilibration equ	877.93	29.23	69.31	15.69	114.23	992.16	40.55	0.00	4.55	87.44	57.51	216.11	406.17	563.48	0.00	563.48	969.64
End of reflood 219.90	877.93	8.79	31.66	15.69	56.14	934.07	40.55	00.00	4.95	121.09	70.72	276.06	513.36	398.19	00.00	398.19	911.55
End of refill 23.00	877.93	0.00	7.61	15.69	23.3	901.23	17.67	10.04	11.8	155.27	76.73	306.95	578.47	322.19	0.00	322.19	900.65
of End of nt blowdown 23.00	877.93	0.00	7.61	15.69	23.3	901.23	15.55	12.16	11.8	155.27	76.73	306.95	578.47	322.19	0.00	322.19	900.65
Start of transient 0.00	877.93	0.00	0.00	0.00	00.00	877.93	308.07	18.52	24.23	162.54	76.62	287.94	877.93	0.00	0.00	00.00	877.93
Time (s)	In RCS, ACC, steam generator		Decay heat	Heat from secondary	Total added	*** Total Available ***	Reactor coolant	Accumulator	Core stored	Primary metal	Secondary metal	Steam generator	Total contents	Break flow	ECCS spill	Total effluent	*** Total Accountable ***
	Initial Energy	Added Energy					Distribution							Effluent			

TABLE 6.2.1-41

ENERGY BALANCE DOUBLE-ENDED PUMP SUCTION - MAXIMUM SAFETY INJECTION

THIS TABLE HAS BEEN DELETED.

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TABLE 6.2.1-42

ENERGY BALANCE DOUBLE-ENDED HOT LEG GUILLOTINE

		Start of transient	End of blowdown	End of refill
Time (s)	(s)	0.00	24.00	24.00
		П	Energy (million Btu)	
Initial Energy	In RCS, ACC, S GEN	877.93	877.93	877.93
Added Energy	Pumped Injection	0.00	0.00	00:00
	Decay Heat	0.00	8.23	8.23
	Heat from Secondary	0.00	13.26	13.26
	Total Added	0.00	21.48	21.48
Total Available	ailable***	877.93	899.41	899.41
Distribution	Reactor Coolant	308.07	23.12	25.18
	Accumulator	18.52	10.79	8.74
	Core Stored	24.23	8.35	8.35
	Primary Metal	162.54	152.85	152.85
	Secondary Metal	76.62	75.47	75.47
	Steam Generator	287.94	299.32	299.32
	Total Contents	877.93	569.91	569.91
Effluent	Break Flow	0.00	328.92	328.92
	ECCS Spill	0.00	0.00	0.00
	Total Effluent	0.00	328.92	328.92
Total Accountable	ountable***	877.93	898.82	898.82

TABLE 6.2.1-43

SYSTEM PARAMETERS

<u>PARAMETER</u>	<u>VALUE</u>	
Core inlet temperature (includes +4.8°F allowance for instrument error and deadband)	561°F	
Vessel outlet temperature (includes +4.8°F allowance for instrument error and deadband)	625.4°F	
Initial steam generator steam pressure	999 psia	
Assumed maximum containment back pressure	66.7 psia	
Pumped injection, assumed for FROTH:		
Minimum (lb/s)	520	

TABLE 6.2.1-44

THROUGH

TABLE 6.2.1-59

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TABLE 6.2.1-60 SPECTRUM OF SECONDARY SYSTEM PIPE RUPTURES ANALYZED(a)

<u>Case</u>	<u>Size</u>	Type Of Rupture	% Power ^(b)
1	Full	Double-ended	102
2	Full	Double-ended	70
3	Full	Double-ended	30
4	Full	Double-ended	0
5	0.60 ft ²	Double-ended with entrainment	102
6	0.53 ft ²	Double-ended with entrainment	70
7	0.36ft^2	Double-ended with entrainment	30
8	0.20 ft ²	Double-ended with entrainment	0
9	0.33 ft^2	Double-ended without entrainment	102
10	0.32 ft^2	Double-ended without entrainment	70
11	0.22 ft^2	Double-ended without entrainment	30
12	0.10 ft ²	Double-ended without entrainment	0
13	0.86 ft ²	Split rupture	102
14	0.908 ft^2	Split rupture	70
15	0.944 ft^2	Split rupture	30
16	0.40 ft ²	Split rupture	0

a. Case 1 through case 16 have assumed the loss of a diesel.b. % power of 3579 MWt.

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TABLE 6.2.1-61 (SHEET 1 OF 9)

MASS AND ENERGY RELEASE DATA FOR CASE 16 - PEAK CALCULATED CONTAINMENT PRESSURE FOR MSLB

Energy Flowrate Btu/s (E+06) 0.954 0.953 0.952 0.951 0.950 0.950 0.948	0.946 0.945 0.945 0.943 0.942 0.940 0.939 0.939	0.938 0.938 0.936 0.935 0.935 0.932 0.930 0.928 0.927 0.927 0.927 0.929 0.919
Mass Flowrate lbm/s 799.79 798.43 797.53 797.05 795.86 795.70 793.65 792.95	792.33 791.61 791.16 790.40 788.93 788.56 787.66 787.82 786.59	785.80 785.80 785.75 784.42 783.92 783.12 782.74 780.51 779.34 777.29 776.01 774.70 774.70 774.70 774.70 774.70 774.70
Time (s) 18.60 18.80 19.00 19.20 19.40 19.60 20.00	20.40 20.60 20.80 21.20 21.40 22.20 22.20 22.40	23.00 23.20 23.40 23.40 24.00 24.00 25.40 26.40 27.00 28.20 28.80 30.40 31.40
Energy Flowrate Btu/s (E+06) 1.005 1.003 1.000 0.999 0.997 0.997	0.994 0.993 0.991 0.989 0.987 0.982 0.980 0.979	0.977 0.977 0.975 0.973 0.973 0.968 0.968 0.968 0.964 0.964 0.965 0.960 0.960
Mass Flowrate Jbm/s 843.43 842.21 841.85 841.43 839.56 838.11 836.71	834.53 833.05 831.83 831.34 829.79 827.84 824.87 822.96 822.96 822.04 821.04	818.88 817.85 816.40 815.61 815.61 812.69 811.88 811.45 810.44 809.04 808.20 806.87 804.57
Time (s) 9.40 9.40 9.60 9.80 10.00 10.20 10.40 10.60	11.20 11.40 11.80 12.20 13.20 13.20 13.60 14.00	14.20 14.40 14.40 14.80 15.00 15.20 15.80 16.00 16.60 16.80 17.20 17.40
Energy Flowrate Btu/s (E+06) 0.000 1.077 1.075 1.073 1.070 1.068 1.066	1.062 1.061 1.059 1.057 1.054 1.052 1.050 1.048 1.046 1.045	1.039 1.036 1.035 1.034 1.033 1.029 1.026 1.025 1.027 1.019 1.019 1.019 1.019 1.019
Mass Flowrate lbm/s	892.96 892.40 890.84 888.62 886.71 885.95 884.39 882.89 880.80 879.46 877.29	873.61 873.61 870.88 869.46 867.83 865.86 864.34 861.82 860.07 858.87 855.98 855.98 855.36 855.36 856.33
Time (s) 0.00 0.20 0.40 0.60 0.80 1.20 1.40	1.80 2.00 2.20 2.40 2.80 3.00 3.20 3.40 4.00 4.00	5.20 5.20 5.20 5.20 5.40 5.80 6.40 6.80 7.20 7.20 7.20 8.00

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TABLE 6.2.1-61 (SHEET 2 OF 9)

Energy Flowrate Btu/s (E+06) 0.907 0.889 0.883 0.609 0.607	0.604 0.602 0.601 0.599 0.594 0.594 0.592	0.590 0.588 0.587 0.585 0.584 0.581 0.580	0.576 0.573 0.573 0.571 0.570 0.567 0.565 0.564	0.560 0.559 0.557 0.557 0.552 0.550 0.549 0.547
Mass Flowrate bm/s 759.23 749.16 743.90 738.67 506.00 504.21	502.44 500.70 499.84 498.13 496.45 493.98 492.37 491.57	489.99 488.43 486.14 485.39 482.44 481.71 480.28	478.87 477.48 476.11 474.76 472.12 471.47 468.31	465.25 464.06 462.31 460.60 458.92 457.28 456.20 454.62
Time (s) 34.80 39.00 41.20 43.40 128.40 128.80	129.20 129.60 129.80 130.20 130.60 131.20 131.80	132.20 132.60 132.80 133.20 133.40 134.20 134.40	135.20 135.60 136.00 136.40 137.20 137.20 137.80 138.40	139.40 139.40 139.80 140.40 141.00 142.20 142.60 143.20
Energy Flowrate Btu/s (E+06) 0.958 0.957 0.956 0.955 0.695	0.691 0.689 0.685 0.684 0.682 0.677	0.675 0.674 0.672 0.669 0.667 0.664 0.664	0.661 0.659 0.658 0.655 0.653 0.651 0.651	0.644 0.641 0.641 0.638 0.637 0.632 0.629 0.629
Mass Flowrate lbm/s 803.29 801.91 801.00 800.77 578.27 576.70	575.15 573.61 572.09 570.59 569.10 566.17 564.73	561.90 560.50 559.12 557.76 556.40 553.74 553.74 552.43	549.85 548.58 547.33 544.85 542.42 541.22 540.04	535.40 533.14 530.92 529.83 527.67 525.54 523.45 521.39
Time (s) 17.80 18.00 18.20 18.40 116.20	116.60 116.80 117.00 117.20 117.40 117.80 118.00 118.20	118.40 118.60 119.00 119.40 119.80 119.80	120.20 120.40 120.60 121.00 121.20 121.40 121.60 121.60	123.00 123.00 123.40 123.40 124.00 124.40 125.20 125.20
Energy Flowrate Btu/s (E+06) 1.011 1.010 1.009 1.007 0.877	0.865 0.860 0.854 0.848 0.837 0.832 0.832	0.821 0.815 0.805 0.799 0.794 0.789	0.779 0.764 0.769 0.759 0.750 0.740	0.733 0.731 0.727 0.723 0.718 0.716 0.715 0.710
Mass Flowrate lbm/s 848.89 847.83 847.27 845.58 733.49 728.34	723.23 718.64 713.63 708.68 703.79 698.94 694.74 689.99	685.30 680.65 676.05 671.49 669.43 666.98 662.52 658.10	649.79 645.51 641.26 637.06 632.91 629.16 625.09 617.06 613.12	609.21 605.70 601.89 600.17 598.12 596.09 593.24 591.49
Time (s) 8.40 8.60 8.80 9.00 45.60	50.00 52.00 54.20 56.40 58.60 60.80 62.60 65.00	67.20 69.40 71.60 74.80 76.00 78.20 80.40	84.60 86.80 89.00 93.40 95.40 97.60 102.00	106.40 108.40 110.60 111.60 114.00 114.40 114.60

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TABLE 6.2.1-61 (SHEET 3 OF 9)

Energy Flowrate Btu/s	(E+06) 0.544	0.543	0.541	0.540	0.538	0.537		0.466	0.466	0.465	0.465	0.465	0.465	0.464	0.464	0.464	0.464	0.464	0.463	0.463	0.463	0.463	0.463	0.463	0.463	0.463	0.463	0.463	0.463	0.463	0.463	0.463	0.463	0.463	0.464	0.464	0.464	0.466	0.467	0.467	0.467	0.467
Mass Flowrate	lbm/s 452.06	451.06	449.59	448.63	447.21	445.82		387.01	386.69	386.41	386.15	385.93	385.72	385.52	385.34	385.19	385.04	384.93	384.81	384.71	384.62	384.54	384.47	384.36	384.28	384.22	384.18	384.16	384.15	384.16	384.19	384.23	384.29	384.41	385.10	385.36	385.61	387.24	387.41	387.51	387.59	387.66
	Time (s)	144.60	145.20	145.60	146.20	146.80		218.80	221.00	223.20	225.40	227.40	229.60	231.80	234.00	236.20	238.40	240.40	242.60	244.80	247.00	249.20	251.40	255.60	260.00	264.20	268.60	273.00	277.60	283.80	290.00	296.00	302.20	314.60	376.40	401.00	425.80	592.20	611.40	623.80	636.00	648.40
Energy Flowrate Btu/s	(E+06) 0.623	0.621	0.620	0.618	0.615	0.613	0.611	0.491	0.490	0.489	0.488	0.487	0.486	0.486	0.485	0.485	0.484	0.483	0.482	0.482	0.481	0.481	0.481	0.480	0.479	0.478	0.478	0.477	0.476	0.476	0.475	0.475	0.474	0.474	0.473	0.473	0.472	0.472	0.471	0.471	0.470	0.470
Mass Flowrate	<u>lbm/s</u> 518.36	516.38	515.40	513.46	511.55	89.605	507.83	407.41	406.86	406.32	405.44	404.60	403.79	403.32	402.85	402.40	401.96	401.25	400.56	400.16	399.77	399.38	399.01	398.40	397.82	397.15	396.62	396.10	395.61	395.03	394.49	394.05	393.64	393.15	392.69	392.33	391.84	391.65	391.13	390.82	390.52	390.18
	Time (s)	126.20	126.40	126.80	127.20	127.60	128.00	172.20	172.80	173.40	174.40	175.40	176.40	177.00	177.60	178.20	178.80	179.80	180.80	181.40	182.00	182.60	183.20	184.20	185.20	186.40	187.40	188.40	189.40	190.60	191.80	192.80	193.80	195.00	196.20	197.20	198.60	198.80	200.40	201.40	202.40	203.60
Energy Flowrate Btu/s	(E+06) 0.708	0.706	0.704	0.702	0.700	0.698	969.0	0.535	0.534	0.532	0.531	0.529	0.528	0.527	0.525	0.524	0.523	0.522	0.521	0.519	0.518	0.517	0.516	0.515	0.514	0.513	0.512	0.511	0.510	0.509	0.508	0.507	0.506	0.505	0.504	0.504	0.503	0.502	0.501	0.500	0.499	0.499
Mass Flowrate	<u>S/mql</u>	588.06	586.38	584.72	583.08	581.46	579.85	444.46	443.57	442.26	440.97	439.71	438.48	437.28	436.10	435.33	434.19	433.45	432.35	431.27	430.22	429.19	428.52	427.52	426.87	425.91	425.28	424.35	423.75	422.85	421.98	421.12	420.55	419.72	418.90	418.11	417.32	416.56	415.81	415.32	414.60	414.13
	Time (s)	115.00	115.20	115.40	115.60	115.80	116.00	147.40	147.80	148.40	149.00	149.60	150.20	150.80	151.40	151.80	152.40	152.80	153.40	154.00	154.60	155.20	155.60	156.20	156.60	157.20	157.60	158.20	158.60	159.20	159.80	160.40	160.80	161.40	162.00	162.60	163.20	163.80	164.40	164.80	165.40	165.80

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TABLE 6.2.1-61 (SHEET 4 OF 9)

Energy Flowrate Btu/s	0.467	0.467	0.467	0.467	0.467	0.467	0.467	0.467	0.467	0.466	0.466	0.401	0.400	0.399	0.398	0.397	0.396	0.394	0.393	0.392	0.390	0.387	0.385	0.383	0.381	0.379	0.377	0.375	0.373	0.370	0.368	0.365	0.364	0.361	0.357	0.355	0.352	0.348	0.338	0.334	0.330	0.327	0.325
Mass Flowrate	387.69	387.72	387.73	387.71	387.69	387.66	387.56	387.52	387.40	387.25	387.07	333.01	332.11	331.20	330.28	329.36	328.43	327.50	326.55	325.60	323.68	321.74	319.77	317.77	316.77	314.74	312.69	311.66	309.59	307.50	305.39	303.26	302.20	300.06	296.83	294.66	292.49	289.23	280.51	277.25	274.01	271.86	269.72
(a) cwiT	654.60	00'.299	08.989	692.40	00.869	703.80	716.80	720.80	732.20	743.60	755.00	878.80	879.00	879.20	879.40	879.60	879.80	880.00	880.20	880.40	880.80	881.20	881.60	882.00	882.20	882.60	883.00	883.20	883.60	884.00	884.40	884.80	885.00	885.40	886.00	886.40	886.80	887.40	889.00	09.688	890.20	890.60	891.00
Energy Flowrate Btu/s	0.470	0.469	0.469	0.469	0.468	0.468	0.468	0.467	0.467	0.467	0.467	0.446	0.445	0.444	0.443	0.442	0.440	0.440	0.438	0.437	0.435	0.435	0.433	0.432	0.432	0.431	0.430	0.429	0.428	0.428	0.427	0.426	0.425	0.424	0.423	0.423	0.422	0.421	0.420	0.419	0.418	0.417	0.416
Mass Flowrate	389.85	389.59	389.34	389.05	388.78	388.56	388.35	388.11	387.92	387.56	387.35	370.26	369.25	368.22	367.70	366.62	365.53	364.97	363.83	362.67	361.47	360.87	359.63	359.00	358.36	357.72	357.07	356.41	355.74	355.07	354.39	353.70	353.00	352.30	351.58	350.86	350.14	349.40	348.65	347.90	347.14	346.38	345.60
Timo (e)	204.80	205.80	206.80	208.00	209.20	210.20	211.20	212.40	213.40	215.40	216.60	868.00	868.40	868.80	869.00	869.40	08.698	870.00	870.40	870.80	871.20	871.40	871.80	872.00	872.20	872.40	872.60	872.80	873.00	873.20	873.40	873.60	873.80	874.00	874.20	874.40	874.60	874.80	875.00	875.20	875.40	875.60	875.80
Energy Flowrate Btu/s	0.498	0.497	0.496	0.495	0.495	0.494	0.494	0.493	0.493	0.492	0.491	0.466	0.466	0.465	0.465	0.465	0.464	0.464	0.463	0.463	0.463	0.463	0.463	0.463	0.463	0.462	0.462	0.462	0.462	0.461	0.461	0.461	0.461	0.460	0.460	0.460	0.459	0.459	0.459	0.458	0.458	0.458	0.457
Mass Flowrate	413.43	412.75	412.08	411.42	410.99	410.36	409.95	409.34	408.94	408.36	407.97	386.88	386.66	386.42	386.18	385.91	385.63	385.34	384.69	384.68	384.62	384.52	384.41	384.27	384.11	383.95	383.77	383.57	383.37	383.16	382.94	382.71	382.46	382.21	381.95	381.68	381.40	381.11	380.81	380.51	380.19	379.87	379.53
in (e)	166.40	167.00	167.60	168.20	168.60	169.20	169.60	170.20	170.60	171.20	171.60	766.20	09.777	789.00	800.20	811.60	823.00	834.40	858.80	859.00	859.20	859.40	859.60	859.80	860.00	860.20	860.40	09.098	860.80	861.00	861.20	861.40	861.60	861.80	862.00	862.20	862.40	862.60	862.80	863.00	863.20	863.40	863.60

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TABLE 6.2.1-61 (SHEET 5 OF 9)

Energy Flowrate Btu/s	(E+06)	0.322	0.319	0.318	0.316	0.313	0.300	0.308	0.307	0.306	0.305	0.303	0.302	0.301	0.300	0.223	0.222	0.222	0.221	0.221	0.221	0.220	0.220	0.219	0.219	0.218	0.218	0.217	0.217	0.217	0.216	0.216	0.215	0.215	0.214	0.214	0.214	0.213	0.213	0.213	0.212	0.211	0.211	0.210	
Mass Flowrate	s/mql	267.39	762.47	264.42	262.33	260.25	258.19	256.15	255.13	254.12	253.12	252.12	251.13	250.14	249.16	185.77	185.37	184.97	184.57	184.18	183.79	183.41	183.03	182.66	182.29	181.92	181.56	181.20	180.85	180.50	180.16	179.81	179.48	179.14	178.81	178.49	178.16	177.85	177.53	177.22	176.91	176.31	176.01	175.43	
	Time (s)	891.40	891.80	892.00	892.40	892.80	893.20	893.60	893.80	894.00	894.20	894.40	894.60	894.80	895.00	913.80	914.00	914.20	914.40	914.60	914.80	915.00	915.20	915.40	915.60	915.80	916.00	916.20	916.40	916.60	916.80	917.00	917.20	917.40	917.60	917.80	918.00	918.20	918.40	918.60	918.80	919.20	919.40	919.80	
Energy Flowrate Btu/s	(E+06)	0.415	0.414	0.413	0.412	0.411	0.410	0.409	0.408	0.407	0.406	0.405	0.404	0.403	0.402	0.253	0.252	0.251	0.251	0.250	0.249	0.248	0.248	0.247	0.246	0.245	0.245	0.244	0.243	0.242	0.242	0.241	0.240	0.239	0.238	0.238	0.237	0.236	0.236	0.235	0.235	0.234	0.233	0.233	
Mass Flowrate	<u>lbm/s</u>	344.82	344.02	343.22	342.42	341.60	340.78	339.94	339.10	338.26	337.40	336.54	335.67	334.79	333.90	210.60	209.93	209.28	208.63	207.99	207.35	206.72	206.10	205.48	204.87	204.27	203.67	203.08	202.49	201.91	201.34	200.77	200.20	199.09	198.54	198.00	197.46	196.93	196.40	195.88	195.37	194.86	194.35	193.85	
	Time (s)	8/6.00	8/6.20	876.40	8/6.60	8/6.80	00.778	877.20	877.40	877.60	877.80	878.00	878.20	878.40	878.60	904.40	904.60	904.80	905.00	905.20	905.40	905.60	905.80	00.906	906.20	906.40	09.906	08.906	902.00	907.20	907.40	09.706	907.80	908.20	908.40	09.806	08.806	00.606	909.20	909.40	09.606	08.606	910.00	910.20	
Energy Flowrate Btu/s	(E+06)	0.457	0.456	0.456	0.455	0.455	0.433	0.454	0.453	0.452	0.451	0.450	0.449	0.448	0.447	0.299	0.297	0.296	0.295	0.294	0.293	0.292	0.291	0.289	0.288	0.287	0.286	0.285	0.284	0.283	0.282	0.281	0.280	0.279	0.278	0.277	0.276	0.275	0.274	0.273	0.272	0.271	0.270	0.269	
Mass Flowrate	lbm/s	3/9.19	3/8.84	378.48	3/8.11	3//./4	577.50	3/6.5/	375.75	374.90	374.47	373.58	372.66	371.72	370.75	248.18	247.22	246.25	245.30	244.35	243.41	242.47	241.54	240.62	239.71	238.80	237.90	237.00	236.11	235.23	234.36	233.49	232.63	231.78	230.93	230.09	229.26	228.43	227.62	226.80	226.00	225.20	224.41	223.62	
	Time (s)	863.80	864.00	864.20	864.40	864.60	804.80	865.20	865.60	866.00	866.20	09.998	867.00	867.40	867.80	895.20	895.40	895.60	895.80	896.00	896.20	896.40	09.968	896.80	897.00	897.20	897.40	09.768	897.80	898.00	898.20	898.40	09.868	898.80	899.00	899.20	899.40	09.668	08.668	00.006	900.20	900.40	09.006	900.80	

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																																												01/10 0/10
Energy Flowrate Rtu/s	(E+06)	0.20	0.209	0.208	0.208	0.207	0.207	0.206	0.206	0.205	0.205	0.204	0.204	0.204	0.203	0.203	0.202	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.181	0.181	0.180	0.179	0.178	0.177	0.176	0.174	0.173	0.172	
Macc Flowrate		174 57	174.02	173.75	173.22	172.70	172.45	172.19	171.70	171.21	170.98	170.51	170.05	169.83	169.39	168.96	168.54	151.86	151.83	151.82	151.80	151.78	151.76	151.75	151.74	151.73	151.74	151.78	152.30	152.41	152.36	152.17	151.84	151.40	150.84	150.19	149.44	148.61	147.70	146.72	145.69	144.58	143.43	
	<u>Time (s)</u>	920.40	920.80	921.00	921.40	921.80	922.00	922.20	922.60	923.00	923.20	923.60	924.00	924.20	924.60	925.00	925.40	1001.60	1004.40	1007.40	1010.00	1015.80	1021.40	1027.20	1038.40	1070.80	1800.00	1800.20	1800.40	1800.60	1800.80	1801.00	1801.20	1801.40	1801.60	1801.80	1802.00	1802.20	1802.40	1802.60	1802.80	1803.00	1803.20	
Energy Flowrate Rtu/s	(E+06)	0.232	0.231	0.230	0.230	0.229	0.229	0.228	0.228	0.227	0.226	0.226	0.225	0.225	0.224	0.224	0.223	0.188	0.187	0.187	0.187	0.187	0.186	0.186	0.186	0.186	0.186	0.185	0.185	0.185	0.185	0.185	0.185	0.185	0.184	0.184	0.184	0.184	0.184	0.184	0.184	0.184	0.184	
Macc Flowrate	103 36	192.87	192.39	191.91	191.44	190.97	190.51	190.05	189.60	189.15	188.71	188.27	187.84	187.41	187.02	186.60	186.18	156.66	156.42	156.24	156.07	155.86	155.70	155.51	155.32	155.18	155.05	154.89	154.73	154.62	154.48	154.34	154.21	154.12	154.03	153.91	153.83	153.72	153.62	153.48	153.40	153.31	153.25	
	<u>Time (s)</u>	910.40	910.80	911.00	911.20	911.40	911.60	911.80	912.00	912.20	912.40	912.60	912.80	913.00	913.20	913.40	913.60	945.00	945.80	946.40	947.00	947.80	948.40	949.20	950.00	950.60	951.20	952.00	952.80	953.40	954.20	955.00	955.80	956.40	957.00	957.80	958.40	959.20	00.096	961.20	962.00	962.80	963.40	
Energy Flowrate Rtii/s	(E+06)	0.262	0.266	0.265	0.264	0.263	0.262	0.261	0.261	0.260	0.259	0.258	0.257	0.256	0.255	0.255	0.254	0.202	0.201	0.201	0.200	0.200	0.199	0.199	0.199	0.198	0.198	0.198	0.197	0.197	0.197	0.196	0.196	0.196	0.195	0.195	0.195	0.194	0.194	0.193	0.193	0.193	0.193	
Mass Flowrate	<u>s/mql</u>	222.07	221.30	220.54	219.79	219.04	218.30	217.57	216.84	216.12	215.41	214.70	214.00	213.31	212.62	211.94	211.26	168.13	167.93	167.53	167.15	166.77	166.40	166.04	165.69	165.35	165.18	164.85	164.53	164.37	164.06	163.76	163.61	163.32	163.03	162.75	162.48	162.22	161.83	161.46	161.22	160.98	160.75	
	Time (s)	901.20	901.40	901.60	901.80	902.00	902.20	902.40	902.60	902.80	903.00	903.20	903.40	903.60	903.80	904.00	904.20	925.80	926.00	926.40	926.80	927.20	927.60	928.00	928.40	928.80	929.00	929.40	929.80	930.00	930.40	930.80	931.00	931.40	931.80	932.20	932.60	933.00	933.60	934.20	934.60	935.00	935.40	

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Energy Flowrate Btu/s	(E+06) 0.170	0.169	0.166	0.164	0.162	0.160	0.158	0.156	0.154	0.151	0.149	0.147	0.145	0.143	0.142	0.140	0.138	0.136	0.134	0.132	0.037	0.036	0.036	0.035	0.035	0.034	0.034	0.033	0.033	0.032	0.032	0.032	0.031	0.031	0.030	0.030	0.030	0.029	0.029	0.029	0.028	0.028	0.028
Mass Flowrate	<u>lbm/s</u> 142.22	140.93	139.04	137.19	135.43	133.68	131.94	130.22	128.51	126.82	125.14	123.48	121.84	120.22	118.61	117.03	115.46	113.92	112.39	110.89	31.41	30.99	30.57	30.15	29.75	29.35	28.95	28.57	28.19	27.83	27.47	27.11	26.77	26.44	26.11	25.79	25.49	25.19	24.89	24.59	24.30	24.00	23.71
	<u>Time (s)</u> 1803.40	1803.60	1803.80	1804.00	1804.20	1804.40	1804.60	1804.80	1805.00	1805.20	1805.40	1805.60	1805.80	1806.00	1806.20	1806.40	1806.60	1806.80	1807.00	1807.20	1826.80	1827.00	1827.20	1827.40	1827.60	1827.80	1828.00	1828.20	1828.40	1828.60	1828.80	1829.00	1829.20	1829.40	1829.60	1829.80	1830.00	1830.20	1830.40	1830.60	1830.80	1831.00	1831.20
Energy Flowrate Btu/s	(E+06) 0.183	0.183	0.183	0.183	0.183	0.183	0.183	0.183	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.182	0.071	0.070	690.0	890.0	0.068	0.067	990.0	0.065	0.064	0.063	0.062	0.062	0.061	090'0	0.059	0.059	0.058	0.057	0.057	0.056	0.053	0.052	0.051
Mass Flowrate	<u>lbm/s</u> 153.12	153.00	152.89	152.79	152.70	152.62	152.54	152.47	152.40	152.35	152.29	152.24	152.19	152.15	152.12	152.05	152.00	151.95	151.91	151.88	60.30	59.52	58.75	58.00	57.25	56.52	55.80	55.10	54.40	53.72	53.04	52.38	51.73	51.09	50.46	49.85	49.24	48.66	48.08	47.49	45.15	43.97	43.39
	<u>Time (s)</u> 964.80	966.20	09.796	00.696	970.40	971.80	973.20	974.60	00.976	977.40	979.00	980.40	981.80	983.20	984.60	987.40	990.20	993.00	00.966	08.866	1816.60	1816.80	1817.00	1817.20	1817.40	1817.60	1817.80	1818.00	1818.20	1818.40	1818.60	1818.80	1819.00	1819.20	1819.40	1819.60	1819.80	1820.00	1820.20	1820.40	1821.20	1821.60	1821.80
Energy Flowrate Btu/s	(E+06) 0.192	0.192	0.192	0.191	0.191	0.191	0.190	0.190	0.190	0.190	0.190	0.189	0.189	0.189	0.189	0.189	0.188	0.188	0.188	0.188	0.130	0.129	0.127	0.125	0.124	0.122	0.120	0.119	0.117	0.116	0.114	0.113	0.1111	0.110	0.108	0.107	0.106	0.104	0.103	0.102	0.100	0.099	860.0
Mass Flowrate	<u>lbm/s</u> 160.53	160.21	159.89	159.69	159.49	159.21	159.02	158.84	158.66	158.49	158.24	158.00	157.84	157.69	157.54	157.39	157.18	157.05	156.92	156.79	109.42	107.97	106.55	105.14	103.76	102.39	101.05	99.73	98.45	97.17	95.92	94.69	93.48	92.28	91.11	96.68	88.83	87.72	86.64	85.57	84.53	83.46	82.42
	<u>Time (s)</u> 935.80	936.40	937.00	937.40	937.80	938.40	938.80	939.20	939.60	940.00	940.60	941.20	941.60	942.00	942.40	942.80	943.40	943.80	944.20	944.60	1807.40	1807.60	1807.80	1808.00	1808.20	1808.40	1808.60	1808.80	1809.00	1809.20	1809.40	1809.60	1809.80	1810.00	1810.20	1810.40	1810.60	1810.80	1811.00	1811.20	1811.40	1811.60	1811.80

TABLE 6.2.1-61 (SHEET 8 OF 9)

Energy Flowrate Btu/s	(E+06)	0.027	0.027	0.026	0.026	0.026	0.025	0.025	0.025	0.024	0.024	0.024	0.023	0.023	0.023	0.022	0.022	0.021	0.021	0.021	0.021	0.020	0.020																				
Mass Flowrate		23.13	22.84	22.55	22.30	22.06	21.81	21.47	21.17	20.89	20.62	20.35	20.08	19.82	19.56	19.04	18.79	18.53	18.26	18.00	17.73	17.46	17.19																				
	<u>Time (s)</u>	1831.60	1831.80	1832.00	1832.20	1832.40	1832.60	1832.80	1833.00	1833.20	1833.40	1833.60	1833.80	1834.00	1834.20	1834.60	1834.80	1835.00	1835.20	1835.40	1835.60	1835.80	1836.00																				
Energy Flowrate Btu/s	(E+06) 0.050	0.050	0.049	0.048	0.048	0.047	0.046	0.046	0.045	0.044	0.044	0.043	0.043	0.042	0.042	0.041	0.041	0.040	0.039	0.039	0.038	0.038	0.037																				
Mass Flowrate	<u>lbm/s</u>	42.23	41.66	41.11	40.55	40.00	39.46	38.92	38.39	37.87	37.54	37.13	36.71	36.28	35.81	35.37	34.93	34.49	33.60	33.16	32.71	32.28	31.84																				
	Time (s)	1822.20	1822.40	1822.60	1822.80	1823.00	1823.20	1823.40	1823.60	1823.80	1824.00	1824.20	1824.40	1824.60	1824.80	1825.00	1825.20	1825.40	1825.80	1826.00	1826.20	1826.40	1826.60																				
Energy Flowrate Btii/s	(E+06) 0 097	0.095	0.094	0.093	0.092	0.090	0.089	0.088	0.087	980.0	0.085	0.084	0.082	0.081	0.080	0.079	0.078	0.077	0.076	0.075	0.074	0.073	0.072	0.020	0.019	0.019	0.019	0.018	0.018	0.018	0.017	0.017	0.016	0.016	0.016	0.015	0.015	0.014	0.014	0.014	0.013	0.013	0.012
Mass Flowrate	lbm/s 81 38	80.35	79.34	78.32	77.32	76.33	75.34	74.37	73.40	72.45	71.51	70.58	99.69	68.75	67.85	96.99	60.99	65.22	64.37	63.53	62.71	61.89	61.09	16.92	16.64	16.35	16.06	15.77	15.47	15.17	14.86	14.55	14.24	13.91	13.59	13.25	12.91	12.57	12.22	11.86	11.49	11.12	10.73
	<u>Time (s)</u>	1812.20	1812.40	1812.60	1812.80	1813.00	1813.20	1813.40	1813.60	1813.80	1814.00	1814.20	1814.40	1814.60	1814.80	1815.00	1815.20	1815.40	1815.60	1815.80	1816.00	1816.20	1816.40	1836.20	1836.40	1836.60	1836.80	1837.00	1837.20	1837.40	1837.60	1837.80	1838.00	1838.20	1838.40	1838.60	1838.80	1839.00	1839.20	1839.40	1839.60	1839.80	1840.00

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TABLE 6.2.1-61 (SHEET 9 OF 9)

Energy Flowrate Btu/s	(E+06)										
Mass Flowrate	s/wal										
	Ime (s)										
Energy Flowrate Btu/s	(E+06)										
Mass Flowrate	<u>s/wal</u>										
į	IIme (s)										
Energy Flowrate Btu/s	(E+06) 0.012	0.011	0.011	0.010	0.010	0.009	0.009	0.008	0.007	0.000	0.000
Mass Flowrate	10.34	9.93	9.51	80.6	8.62	8.13	7.59	86.9	6.17	0.00	0.00
į	<u>lime (s)</u> 1840.20	1840.40	1840.60	1840.80	1841.00	1841.20	1841.40	1841.60	1841.80	1842.00	1900.00

TABLE 6.2.1-62 (SHEET 1 OF 8)

MASS AND ENERGY RELEASE DATA FOR CASE 13 - PEAK CALCULATED CONTAINMENT TEMPERATURE FOR MSLB

Energy Flowrate Btu/s (E+06)	VANAVANAVANAVANAVANAVANAVANAVANAVANAVAN
Mass Flowrate Ibm/s	### ### ##############################
Time (s)	22444724244444444444444444444444444444
Energy Flowrate Btu/s (E+06)	トートートートートートートーーイン
Mass Flowrate <u>lbm/s</u>	ただいたいではいるであるであるであるであるできたとしてアンドンドンドンドンドンドンドンドンドンドンドンドンドンドンドンドンドンドンド
Time (<u>s)</u>	55-1-1-1-1997994444444440000000000000000000000000
Energy Flowrate Btu/s (E+06)	0. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0
Mass Flowrate <u>lbm/s</u>	C4444444444444444444444444444444444444
Time <u>(s)</u>	00000

	Energy Flowrate Btu/s (E+06)	
	Mass Flowrate <u>lbm/s</u>	x_0
	Time (<u>s)</u>	\$\$\$\$\$\$5-5-5-5-5-5-5-5-5-5-5-5-5-5-5-5-5
	Energy Flowrate Btu/s (E+06)	
))	Mass Flowrate <u>lbm/s</u>	$\begin{array}{c} www. www. www. www. www. www. www. ww$
	Time (<u>s)</u>	KKKKKKK KKKKKKKKKKKKKKKKKKKKKKKKKKKKKKKKKKKK
	Energy Flowrate Btu/s (E+06)	
	Mass Flowrate <u>lbm/s</u>	66666744446666666666666666666666666666
	Time (<u>s)</u>	4002PPQQQQQQQQQQQQQQQQQQQQQQQQQQQQQQQQQQ

	000000	0000000		0000000	0000000	0000000	0000000	0000
Mass Flowrate Ibm/s	444444 0000000 0000000	4444444 9000000000000000000000000000000	4444444 9000000000000000000000000000000	4444444 9000000000000000000000000000000	. 4444444 19444444 1999999999999999999999	4444444 00000000 000000000000000000000	. 44444444 . 94444444 . 9000000000 . 900000000000000000000000	24444 20000 20000
Time (<u>s)</u>	00000000000000000000000000000000000000	9,00,00,00,00,00,00,00,00,00,00,00,00,00	00000000000000000000000000000000000000	00000000000000000000000000000000000000	00000000 00000000 00000000	00000000000000000000000000000000000000	60000000000000000000000000000000000000	734.0 738.0 738.0 740.0
Energy Flowrate Btu/s (E+06)							00000000000000000000000000000000000000	
Mass Flowrate Ibm/s	444444 90000000 0000000	4444444 9000000000000000000000000000000	- 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	2444444 2000000000000000000000000000000	4444444 2000000000000000000000000000000	4444444 000000000000000000000000000000	4444444 2000000000000000000000000000000	744.8 49.8
Time (<u>s)</u>	0000000	გ. გ.გ.გ.გ.გ.გ.გ.გ. ი.გ.ე.გ.გ.გ.გ.გ. ე.ე.ე.ე.ე.ე.ე.	00000000000000000000000000000000000000	0.000,000,000,000,000,000,000,000,000,0	00000000000000000000000000000000000000	00000000000000000000000000000000000000	00000000000000000000000000000000000000	634.0 634.0
Energy Flowrate Btu/s (E+06)	000000 1177777 1777777 13333333	00000000 	0000000 	000000 	0000000 1000000 1000000 10000000000000	000000 000000 000000 00000000000000000	00000000000000000000000000000000000000	0.1774 0.1774
Mass Flowrate Ibm/s	774744 444444 0000000 ∞∞∞∞∞∞	4444444 9000000000000000000000000000000	- 4444444 9000000000000000000000000000000	444444 90000000000000000000000000000000	444444 90000000000000000000000000000000	444444 90000000 000000000	4444444 9000000000000000000000000000000	944 994 999
Time (<u>s)</u>	444444 %%%%% %0000000 0000000	444440000 444440000 74000040 500000000	1444444 20000000000000000000000000000000	44444 777788 00248800 0000000	444444 88880000 140800146 00000000	40000000000000000000000000000000000000	127272727272727272727272727272727272727	530.0 530.0

Energ Flowrs Btu/s (E+0	000000000000000000000000000000000000000
Mass Flowrate <u>Ibm/s</u>	44444444444444444444444444444444444444
Time (<u>s)</u>	@@@@@@@@@@@@@@@@@@@@@@@@@@@@@@@@@@@@@@
Energy Flowrate Btu/s (E+06)	00000000000000000000000000000000000000
Mass Flowrate <u>lbm/s</u>	44444444444444444444444444444444444444
Time (<u>s)</u>	$\begin{array}{c} \alpha$
Energy Flowrate Btu/s (E+06)	00000000000000000000000000000000000000
Mass Flowrate <u>lbm/s</u>	444444444444444444444444444444444444
Time (S)	744446000000000000000000000000000000000

	Mass Flowrate <u>Ibm/s</u>	444444444444444444444444444444444444
	Time (<u>s)</u>	77777777777777777777777777777777777777
	Energy Flowrate Btu/s (E+06)	COOCOOCOOCOOCOOCOOCOOCOOCOOCOOCOOCOOCOO
6.2.1-62 (SHEET 6 OF 8)	Mass Flowrate <u>Ibm/s</u>	44444444444444444444444444444444444444
TABLE 6.2.	Time (<u>s)</u>	### ##################################
	Energy Flowrate Btu/s (E+06)	00000000000000000000000000000000000000
	Mass Flowrate <u>lbm/s</u>	44444444444444444444444444444444444444
	Time (<u>s)</u>	00000000000000000000000000000000000000

	Mass Flowrate <u>lbm/s</u>	444444444444444444444444444444444444
	Time (S)	######################################
	Energy Flowrate Btu/s (E+06)	00000000000000000000000000000000000000
TABLE 6.2.1-62 (SHEET 7 OF 8)	Mass Flowrate <u>lbm/s</u>	<u>4444444444444444444444444444444444444</u>
TABLE 6.2.1	Time (<u>s)</u>	44444400000000000000000000000000000000
	Energy Flowrate Btu/s (E+06)	00000000000000000000000000000000000000
	Mass Flowrate <u>lbm/s</u>	444444444444444444444444444444444444
	Time (s)	\$\text{\tex{\tex

Energy Flowrate Btu/s (E+06)

	Mass Flowrate Ibm/s	
	Time (<u>s)</u>	
	Energy Flowrate Btu/s (E+06)	00000000000000000000000000000000000000
6.2.1-62 (SHEET 8 OF 8)	Mass Flowrate Ibm/s	### ### ##############################
TABLE 6.2	Time (s)	### ##################################
	Energy Flowrate Btu/s (E+06)	00000000000000000000000000000000000000
	Mass Flowrate Ibm/s	444444444444444444444444444444444444
	Time (<u>s)</u>	LLLLLLLLLLLLLLLLLLLLLLLLLLLLLLLLLLLL

TABLE 6.2.1-63

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TABLE 6.2.1-64 (SHEET 1 OF 3)

SPECIFIC PLANT DESIGN INPUT FOR MSLB ANALYSIS

TABLE 6.2.1-64 (SHEET 2 OF 3)

Case	7	_∞	6	10	11	12
Initial steam generator inventory (lbm) Faulted	154,700	186,400	115,000	129,000	154,700	186,400
Initial steam pressure (psia)	142,600	1102	1003	1051	1112	1102
Mass added by feedwater pumping (lbm)	11,200	23,300	16,000	18,700	11,600	29,200
Mass added by feedwater flashing (lbm)	22,600	24,900	20,800	21,500	22,600	24,900
Unisolatable steamline volume (ft^3)	470	470	470	470	470	470
Auxiliary feedwater addition rate (lbm/h)	5.48E+05	5.48e+05	5.48E+05	5.48E+05	5.48E+05	5.48E+05
Main steam line isolation time (s)	61.6	280.9	14.2	63.1	92.1	188.1
Main feedwater line isolation signal reached (s)	16.4	112.7	11.2	18.3	22.2	185.1
Termination of auxiliary feedwater addition (s)	1800	1800	1800	1800	1800	1800

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TABLE 6.2.1-64 (SHEET 3 OF 3)

Case		13	41	15	16
Initial steam generator inventory (Ibm)	Faulted Intact	116,000 107,700	130,100 123,500	155,900 142,800	186,300 186,300
Initial steam pressure (psia)		1008	1051	1112	1102
Mass added by feedwater pumping (lbm)		25,100	17,600	10,700	4800
Mass added by feedwater flashing (lbm)		20,800	21,500	22,600	0
Unisolatable steam line volume ($ m ft^3$)		470	470	470	470
Auxiliary feedwater addition rate (lbm/h)		5.41E+05	5.41E+05	5.41E+05	5.47E+05
Main steam line isolation time (s)		50.1	48.0	46.5	114.0
Main feedwater line isolation time (s)		17.1	15.5	14.3	23.8
Termination of auxiliary feedwater addition (s)		1800	1800	1800	1800

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TABLE 6.2.1-65

SUMMARY OF RESULTS FOR MSLB CONTAINMENT PRESSURE - TEMPERATURE ANALYSIS

25.79 psig *at Time (s)	94.5 107.6 129.0	109.0 201.0	409.2 537.4	- 854.1	442.9 1085.7	- 89.7	93.8	96.7 484.6
17.19 psig *at Time (s)	28.6 27.5 31.3	10.5 61.1	80.3 52.1	133.1 136.5	53.6 83.7	40.4	38.2	36.7 103.8
6.29 psig *at Time (s)	2.2.9 0.4.0	6.1 6.4	6.8 4.	17.6 11.0	11.2	33.7	8.5	7.3 16.8
Dryout Time (s)	1822 1822 1834	1808 1844	1864 1912	6283 1898	1920 2155	8600 1832	1834	1856 1842
Max. Vapor Temp *at Time (°F at s)	232 at 125 232 at 157 238 at 197	238 at 173 271 at 107	252 at 107 260 at 95	226 at 119 249 at 111	273 at 71 262 at 97	220 at 135 303 at 109	301 at 109	299 at 107 260 at 117
Max. Press. at *Time (psig at Time)	27.6 at 124 27.5 at 1809 30.4 at 197	30.2 at 173 27.1 at 1808	28.3 at 1833 32.4 at 1820	22.9 at 6339 28.5 at 1831	30.8 at 1836 32.7 at 2177	17.0 at 8603 31.9 at 190	31.3 at 226	31.8 at 327 36.5 at 898
Break Type	Double ended Double ended Double ended	Double ended Split	Split	Split Split				
Break Size (ft²)		Full 0.60	0.53 0.36	0.20	0.32	0.10	0.908	0.944
Power ^(a) Level (%)	102 70 30							
Case No.	− 0 €	4 10	9 /	ထတ	19	2 5	4	15

a. % Power Level of 3579 MWt.

TABLE 6.2.1-66

SEQUENCE OF EVENTS FOR CASE 16 - PEAK CALCULATED CONTAINMENT PRESSURE FOR MSLB

Time (s)	<u>Event</u>	
0.0	Break occurs, blowdown from all steam generators.	
16.8	Containment pressure setpoint for isolation of main feedwater lines reached (6.29 psig).	
23.8	Main feedwater line isolation valves closed.	
103.8	Containment pressure setpoint for isolation of main steam lines reached (17.19 psig).	
114	Main steam line isolation valves closed, blowdown from broken loop steam generator and unisolated steam piping only.	
116.8	Air coolers start.	
117.0	Peak containment vapor temperature of 260°F is reached.	
484.6	Containment pressure setpoint for actuation of containment sprays reached (25.79 psig).	
584.6	Containment sprays start.	
897.6	Peak containment pressure of 36.5 psig is reached.	
1800.0	Auxiliary feedwater addition is terminated.	
1842	Dryout occurs, steam generator dry following auxiliary feedwater termination.	

TABLE 6.2.1-67

SEQUENCE OF EVENTS FOR CASE 13 - PEAK CALCULATED CONTAINMENT TEMPERATURE FOR MSLB

Time (s)	<u>Event</u>
0.0	Break occurs, blowdown from all four steam generators.
10.1	Containment pressure setpoint for isolation of main feedwater lines reached (6.29 psig).
22.0	Main feedwater line isolation valves closed.
110.2	Air coolers start.
40.4	Containment pressure setpoint for isolation of main steam lines reached (17.19 psig).
50.1	Main steam line isolation valves closed, blowdown from broken loop steam generator and unisolated steam piping only.
89.7	Containment pressure setpoint for actuation of containment sprays reached (25.79 psig).
109.0	Peak containment vapor temperature of (303.1°F) is reached.
190.3	Containment sprays start.
190.4	Peak containment pressure of 31.9 psig is reached.
1800.0	Auxiliary feedwater addition is terminated.
1832.0	Dryout occurs, steam generator dry following auxiliary feedwater termination.

Time (s)	Mass Flow Rate (lbm/s)	Energy Release Rate (Btu/s)
0	65184	34575122
1	65184	34575122
2	56193	30344429
3	41256	22657374
4	35492	19926677
5	28425	16498775
6	27812	16227833
7	25996	15413707
8	23030	14325843
9	19958	12961246
10	17589	11668789
11	15609	10428740
12	14522	9583417
13	13691	8944095
14	12766	8368203
15	11960	7867605
16	10654	7135285
17	8188	5941968
18	6797	5107798
19	6620	4742754
20	7356	4747262
21	7008	4062956
22	6965	3625279
23	6952	3234900
24	6121	2631796
25	4973	1945657
26	4443	1644792
27	3827	1333195
28	3141	1068429
29	2673	855329
30	2296	700974
31	1928	548382
32	1605	458978
33	1379	324043
33.39	878	186500

Time (s)	Mass Flow Rate (lbm/s)	Energy Release Rate (Btu/s)
45.9	35	37993
55.9	72	82068
65.9	105	122137
75.9	122	144865
85.9	120	144282
95.9	119	143294
120.9	160	156786
145.9	324	195848
170.9	404	221510
195.9	406	221418
220.9	398	224504
245.9	389	230303
270.9	375	225886
295.9	378	226935

TABLE 6.2.1-70 $\label{eq:BROKEN LOOP INJECTION SPILL DURING BLOWDOWN } (C_D \!\!=\!\! 0.6, LOW \, T_{AVG})$

Time	Mass Flow Rate
(s)	(lbm/s)
0	3138
1	2995
2	2798
3	2637
4	2503
5	2388
6	2288
7	2201
8	2122
9	2051
10	1988
11	1930
12	1877
13	1828
14	1783
15	1741
16	1702
17	1665
18	1631
19	1599
20	1569
21	1541
22	1514
23	1489
24	1466
25	1444
26	1423
27	1402
28	1383
29	1365
30	1347

TABLE 6.2.1-71

ACTIVE HEAT SINKS FOR MINIMUM CONTAINMENT PRESSURE ANALYSIS

Containment Spray Parameters

Number of pumps operating	2
Maximum Spray Flow (gal/min)	6748
Fastest post-LOCA initiation of spray	74.0
pumps, assuming offsite power loss	
and no diesel failure(s)	

Containment Fan Coolers

Number of fan coolers	8
Maximum CCWS flow (gal/min)	8975
Maximum NSCW water flow (gal/min)	9625
Fastest post-LOCA initiation of fan	41.1
coolers assuming offsite power loss	
and no diesel failure(s)	

TABLE 6.2.1-72 (SHEET 1 OF 2)

PASSIVE HEAT SINKS(a)

Wall	Material	Thermal Conductivity (Btu/h-ft-°F)	Volumetric Heat Capacity (Btu/ft³-°F)	Thickness (ft)	Area (ft²)
1	Epoxy Zinc coating Steel Concrete Concrete	3.5 1.5 27.0 0.92 0.92	20.0 20.0 58.8 22.62 22.62	0.00025 0.00020833 0.0208333 0.5 5.5	32340
2	Zinc Coating Steel Concrete Concrete	1.5 27.0 0.92 0.92	20.0 58.8 22.62 22.62	0.00020833 0.020833 0.5 5.5	72043
3	Steel Epoxy Concrete Concrete	27.0 3.5 0.92 0.92	58.8 20.0 22.62 22.62	0.015833 0.001125 0.5 6.3333	11442
4	Epoxy Concrete Concrete Steel Concrete	3.5 0.92 0.92 27.0 0.92	20.0 22.62 22.62 58.8 22.62	0.0015417 0.5 2.25 0.020833 10.5	15086.5
5	Zinc Coating Steel Concrete Concrete	1.5 27.0 0.92 0.92	20.0 58.8 22.62 22.62	0.00020833 0.020833 0.5 7.5	4957
6	Zinc Coating Steel	1.5 27.0	20.0 58.8	0.00020833 0.125	595.7
7	Galvanization Steel	65.0 27.0	41.0 58.8	0.00020833 0.004375	316975
8	Zinc Coating Steel	1.5 27.0	20.0 58.8	0.00020833 0.055	159120.9
9	Epoxy Zinc Coating Steel	3.5 1.5 27.0	20.0 20.0 58.8	0.00025 0.00020833 0.020833	30030

TABLE 6.2.1-72 (SHEET 2 OF 2)

Wall	Material	Thermal Conductivity (Btu/h-ft-°F)	Volumetric Heat Capacity (Btu/ft³-°F)	Thickness (ft)	Area (ft²)
10	Zinc Coating Steel	1.5 27.0	20.0 58.8	0.00020833 0.020833	159134
11	Zinc coating Steel	1.5 27.0	20.0 58.8	0.00020833 0.041667	101266.4
12	Zinc Coating Steel	1.5 27.0	20.0 58.8	0.00020833 0.10417	30474.2
13	Steel	27.0	58.8	0.0083333	75041
14	Steel	27.0	58.8	0.09375	232
15	Steel	27.0	58.8	0.04425	1259.4
16	Epoxy Concrete Concrete	3.5 0.92 0.92	20.0 22.62 22.62	0.001125 0.5 6.8333333	44202
17	Epoxy Concrete Concrete	3.5 0.92 0.92	20.0 22.62 22.62	0.000875 0.5 3.5	19550.3
18	Zinc Coating Steel Concrete Concrete	1.5 27.0 0.92 0.92	20.0 58.8 22.62 22.62	0.00020833 0.083333 0.5 4.0	1546.8
19	Epoxy Concrete Concrete	3.5 0.92 0.92	20.0 22.62 22.62	0.001125 0.5 4.0	2215.4
20 ^(a)	Steel	27.0	58.8	0.0083333	100000

a. Wall 20 is the additional 357,900 lbm of metalmass allowance in the containment which is documented in Westinghouse SECL-99-021, Rev. 2. The effects of this are included in the LB LOCA PCT in table 15.6.5-4.

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FORCES AND MOMENTS FOR BROKEN COLD LEG WITH SUPPORT

TABLE 6.2.1-73

-6.76x10 ⁵ -2.59x10 ⁵ 7.24x10 ⁵ 0	İ	190.56' 17-24 -1.10x10 ⁶ -5.54x10 ⁵ 1.23x10 ⁶ -1.94x10 ⁶	184.64' 25-32 -1.09x10 ⁶ -5x42x10 ⁵ 1.22x10 ⁶	179.06' 33.40 -1.02×10° -5.61×10° 1.63×10° 4.49×10°	172.61' 41-48 -4.96×10 ⁵ -2.43×10 ⁵ 5.52×10 ⁶	Total 1-48 -41.9x10 ⁵ -20.6x10 ⁵ 77.3x10 ⁶
0 0		-1.94x10° 3.84x10 ⁶	1.27×10° -2.56×10 ⁶	4.49x10° -8.15x10 ⁶	3.50x10° -7.14x10°	

TABLE 6.2.1-74

UPLIFT FORCES AND MOMENTS FOR BROKEN COLD LEG WITH SUPPORT

	Loads		
	F _Z	M_{Z}	
<u>Nodes</u>	<u>(lbf)</u>	<u>(ft-lbf)</u>	
1-16	0	-1.27 x 10 ⁵	
17-24	8.45 x 10 ⁵	-1.40 x 10 ³	
25-32	-4.76 x 10 ⁴	-1.40 x 10 ³	
49	4.71 x 10 ⁵	0	
50	2.62 x 10 ⁵	0	
Total	7.40 x 10 ⁵	-1.23 x 10 ⁵	

TABLE 6.2.1-75

DEPS MINIMUM SI BREAK INACTIVE METAL ENERGY

Inactive Metal Location	Energy Release Rate (Btu/s)	Start Time (s)	End Time (s)
Reactor Vessel Upper Head	420.1	23.0	12,600
Pressurizer	34.4	23.0	236,017
Steam Generators	214.0	0.0	86,400

TABLE 6.2.2-1 CONTAINMENT FAN COOLER DESIGN DATA

Characteristic	Quantity
Number of units	8
Containment atmosphere design inlet conditions, post LOCA	
Design temperature, saturation (°F)	270
Design pressure (psia)	60
Steam partial pressure (psia)	42
Air partial pressure (psia)	18
Cooling water (nuclear service cooling water)	
Minimum flowrate (gal/min/unit)	700
nlet temperature post-LOCA (°F)	95
Peak outlet temperature (°F)	257
Design post-LOCA heat removal rate at 270°F containment temperature (Btu/h/unit)	56.3 x 10 ⁶
Fan flowrate (actual ft³/min)	43,500 (slow speed)
Static head (in. WG)	0.85
Motor horsepower	62.5

TABLE 6.2.2-2

CONTAINMENT FAN COOLING HEAT REMOVAL CAPACITY (POST-LOCA MODES)

(NSCW Temperature 40°F)

Containment Temperature (°F)		Capacity (Btu/h)
120 150 180 210 240 270		16.39E+06 25.92E+06 39.65E+06 53.57E+06 66.09E+06 77.97E+06
	(NSCW Temperature 95°F)	
Containment Temperature (°F)		Capacity (Btu/h)
120 145 170 195 220 245 270		5.80E+06 12.96E+06 21.23E+06 29.92E+06 38.76E+06 47.54E+06 56.30E+06

TABLE 6.2.2-3 (SHEET 1 OF 5)

CONTAINMENT COOLING SYSTEM FAILURE MODE AND EFFECTS ANALYSIS

Failure Effect on System Safety Function Capability	None; loss of train A; ^(b) train B available.		None; loss of train A; train B available	None; loss of train A; train B available	None; loss of train A; train B available	None; loss of train A; train B available
Method of Failure <u>Detection</u>	Switchgear alarm Motor indicating lights		Switchgear alarm Motor indicating lights	Sequencer alarm Motor indicating lights	(O)	(O)
Failure <u>Mode(s)</u>	Inadvertent open, one breaker		Inadvertent open, one breaker	Fail to close, one breaker	Fail to operate, one motor	Fail to restart and operate, one motor and fan
Plant Operating <u>Mode^(a)</u>	Þ		∢	ω	∢	Ф
Safety <u>Function</u>	Provide continuity and protection for item 3 motor and fan		Provide continuity and protection for item 3 motor and fan		Provide motive power for circulating air in the CTB	
Description of Component	Breakers for high-speed operation) on 1AB04, 480-V switchgear, 1E bus, train A, normally closed (NC)	No. 04 breaker for A7-001 motor and fan. No. 08 breaker for A7-002 motor and fan. No. 12 breaker for A7-005 motor and fan. No. 16 breaker for A7-006 motor and fan.	Breakers (for low-speed operation) on 1AB04, 480-V switchgear, 1E bus, train A, normally open (NO)	No. 05 breaker for A7-001 motor and fan. No. 09 breaker for A7-002 motor and fan. No. 13 breaker for A7-005 motor and fan. No. 17 breaker for A7-006 motor and fan.	Containment building (CTB) cooling unit motor and fan, train A, normally anardized (NE)	1-1501-A7-005-M01 1-1501-A7-005-M01 1-1501-A7-005-M01 1-1501-A7-006-M01
Item No.	-		8		ю	

TABLE 6.2.2-3 (SHEET 2 OF 5)

Failure Effect on System Safety Function Capability	None; no loss of train A. NO damper will remain open.	None; no loss of train A. NO damper will remain open.	None; loss of train A; train B available	None; no loss of train A. NO damper will remain open.	None; loss of train A; train B available	None; loss of train A; train B available
Method of Failure <u>Detection</u>	Motor control center (MCC) alarm Position indicating lights	MCC alarm Position indicating lights	Position indicating lights	Position indicating lights	Position indicating lights	Position indicating lights
Failure <u>Mode(s)</u>	Inadvertent open, one breaker	Inadvertent open, one breaker	Inadvertent closed, one motor starter	Fail to close	Inadvertent closed, one damper	Inadvertent closed, one damper
Plant Operating <u>Mode^(a)</u>	∢	ω	∢	ω	∢	ω
Safety <u>Function</u>	Provide continuity and protection for item 6, damper		Provide continuity to one damper, item 6		Allow flow of air to the containment building and prevent backflow	
Description of Component	Breakers, 480-V MCC, 1E bus, train A, NC No. 26 breaker on 1ABE for A7-001 damper (HV2582A) No. 27 breaker on 1ABE for	A7-002 damper (HV2582B) No. 07 breaker on 1ABC for A7-005 damper (HV2584A) No. 08 breaker on 1ABC for A7-006 damper (HV2584B)	Motor starters for item 6, train A NO NO. 26 motor starter on	1ABE for A7-001 damper No. 27 motor starter on 1ABE for A7-002 damper No. 07 motor starter on 1ABC for A7-005 damper No. 08 motor starter on 1ABC for A7-006 damper	Motor-operated on-off dampers, train A, NO HV2582A damper on 1ABE	For A7-001 HV2582B damper on 1ABE for A7-002 HV2584A damper on 1ABC for A7-006 for A7-006
Item No.	4		വ		Ø	

TABLE 6.2.2-3 (SHEET 3 OF 5)

Failure Effect on System Safety Function Capability	None; loss of train A; train B available.	None; no loss of train A. NO damper will remain open.	None; loss of train A; train B available.	None; loss of train B; train A available.		None; loss of train B; train A available.	None; loss of train B; train A available.
Method of Failure <u>Detection</u>		(C)		Switchgear alarm Motor indicating lights		Switchgear alarm Motor indicating lights	Sequencer alarm Motor indicating lights
Failure <u>Mode(s)</u>	Inadvertent close, one damper	Fail to close, one damper	Fail to reopen, one damper	Inadvertent open, one breaker		Inadvertent open, one breaker	Fail to close, one breaker
Plant Operating <u>Mode^(a)</u>	∢	Ф		∢		∢	ш
Safety <u>Function</u>	Allow flow of air to the containment building and	prevent backirow		Provide continuity and protection for item 10 motor and fan		Provide continuity and protection for item 10	
Description <u>of Component</u>	001, 003 backflow damper, train A, NO			Breakers, (for high speed operation) on 1BB06, 480-V switchgear, 1E bus, train B, NC	No. 04 breaker for A7-003 motor and damper No. 08 breaker for A7-004 motor and damper No. 12 breaker for A7-007 motor and damper No. 16 breaker for A7-008 motor and damper	Breakers, (for low-speed operation) on 1BB06, 480-V switchgear, 1E bus, train B NO	No. 05 breaker for A7-003 motor and damper No. 09 breaker for A7-004 motor and damper No. 13 breaker for A7-007 motor and damper No. 17 breaker for A7-008 motor and damper
No.	7			ω		O	

TABLE 6.2.2-3 (SHEET 4 OF 5)

Failure Effect on System Safety Function Capability	None; loss of train B; train A available	None; loss of train B; train A available	None; no loss of train B. NO damper	wiii remain open.	None, no loss of	will remain open.	None; loss of train B; train A available		None; no loss of train B. NO damper will remain open.	None; loss of train B; train A available		None; loss of train B; train A available
Method of Failure <u>Detection</u>	(0)	()	MCC alarm	Position Indicating lights	Switchgear alarm	Position indicating lights	Position indicating lights		Position indicating lights	Position indicating lights		Position indicating lights
Failure <u>Mode(s)</u>	Fail to operate, one motor and fan	Fail to restart and operate, one motor and fan	Inadvertent open, one	Dreaker	Inadvertent	breaker	Inadvertent close, one motor	starter	Fail to close	Inadvertent close, one		Inadvertent close, one damper
Plant Operating <u>Mode^(a)</u>	∢	В	∢		В		⋖		ш	∢		Ф
Safety Function	Provide motive power for circulating air in the containment	guilding	Provide continuity and protection for	item is damper			Provide continuity to one damper, item 13			Allow flow of air to the containment	prevent backflow	
Description of Component	CTB cooling unit motor and fan, train B, NE 1-1501-A7-003-M01	1-1501-A7-004-M01 1-1501-A7-008-M01 1-1501-A7-008-M01	Breakers, 480-V MCC, 1E bus, train B, NC	No. 26 breaker on 1BBE for A7-003 damper (HV2583A) No. 27 breaker on 1BBE for A7-004 damper (HV,5833B)	A7-004 damper (117-2333B) No. 07 breaker on 1BBC for A7-007 damper (HV/2585A)	No. 08 breaker on 1BBC for A7-008 damper (HV2585B)	Motor starter for item 13, train B NO	No. 26 motor starter on 1RRF for A7-003 damner	No. 27 motor starter on 1BBE for A7-004 damper No. 07 motor starter on 1BBC for A7-007 damper No. 08 motor starter on 1BBC for A7-008 damper	Motor-operated on-off dampers, train B, NO	HV2583A damper on 1BBE	NY 2583B damper on 1BBE for A7-004 damper on 1BBC HV2585A damper on 1BBC for A7-007 damper HV2585B damper on 1BBC for A7-008 damper
Item No.	10						5			5		

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TABLE 6.2.2-3 (SHEET 5 OF 5)

Failure Effect on System Safety Function Capability	None; loss of train B; train A available.	None; no loss of train B; NO damper will remain open.	None; loss of train B; train A available.	None, loss of train A; train B available	None; loss of train B; train A available	None; loss of train A; train B available	None; loss of train B; train A available
Method of Failure <u>Detection</u>		(C)		Motor indicating light	Motor indicating light	Flow indication Temperature alarm high Leak detector level alarm	Flow indication Temperature alarm high Leak detector level alarm
Failure <u>Mode(s)</u>	Inadvertent close, one damper	Fail to close, one damper	Fail to reopen, one damper	Mechanical failure	Mechanical failure	Leakage in cooling coil	Leakage in cooling coil
Plant Operating <u>Mode^(a)</u>	∢	Ф		∢	∢	∢	∢
Safety <u>Function</u>	Allow flow of air to the containment building and prevent	Dackilow		Provide circulation of air inside containment building	Provide circulation of air inside containment building	Provide cooling and heat removal inside containment building	Provide cooling and heat removal inside containment building
Description of Component	002, 004 backflow damper train B, NO			Fans and dampers for train A: 1-1501-A7-001-000 1-1501-A7-002-000 1-1501-A7-005-000	Fans and dampers for train B: 1-1501-A7-003-000 1-1501-A7-004-000 1-1501-A7-007-000	Cooling coils for train A: 1-1501-A7-001-000 1-1501-A7-002-000 1-1501-A7-006-000	Cooling coils for train B: 1-1501-A7-003-000 1-1501-A7-004-000 1-1501-A7-007-000 1-1501-A7-008-000
ltem No.	4			15	91	17	8

a. A - Normal, four fans in train A or four fans in train B operating; B - safety injection, same as normal except all fans are sequenced at low speed. Four fans required (load shed occurs only under loss of offsite power).
 b. Loss of one breaker or fan degrades the performance of that train. The other train remains available.
 c. Flow indication inferred by motor indicating lights and switchgear alarm.

TABLE 6.2.2-4

CONTAINMENT SPRAY SYSTEM COMPONENT DESIGN PARAMETERS

Containment Spray Pump

Type Horizontal centrifugal

Quantity2Design pressure (psig)300Design temperature (°F)250Design flowrate (gal/min)2600Design head (ft)450

Material Stainless steel

Containment Spray Nozzle

Quantity 342^(a)

Type Spraco 1713A or equivalent

Flow per nozzle at

40 psi Δp (gal/min) 15.2

Material Stainless steel

Refueling Water Storage Tank

Quantity 1

Nominal volume (gal) 715,000
Boric acid concentration (ppm) 2400-2600
Design pressure Hydraulic head

Design temperature (°F) 150

Operating pressure Hydraulic head

Material Stainless steel-lined concrete

Eductors^(b)

Quantity 2

Eductor inlet (spray water)

Operating fluid
Operating temperature
Operating temperature
Design flow (gal/min)
NaOH concentration (wt percent)
Design temperature (°F)
Design pressure (psig)

Borated water
Ambient
39.3
30-32
30-32
300
300

Material Stainless steel

a. On Unit 1, one nozzle is currently obstructed to prevent flow through the nozzle.

b. The spray additive subsystem has been abandoned in place. The eductors currently serve only to maintain the spray system pressure boundary integrity.

TABLE 6.2.2-5 (SHEET 1 OF 2)

FAILURE MODE AND EFFECTS ANALYSIS - CONTAINMENT SPRAY SYSTEM - ACTIVE COMPONENTS

Remarks	Valve is normally closed during power operations. Valve opens on actuation by a CS signal.	Pump circuit breaker is aligned to close on actuation by a CS signal.
Failure <u>Detection Method^(b)</u>	Valve position indication (closed to open position change) at CB. Valve monitor light and alarm (closed position) for group monitoring of components at CB.	Open pump switchgear circuit breaker indication at CB. Circuit breaker close position monitor light and alarm for group monitoring of components at CB common breaker trip alarm at CB.
Effect on System Operation ^(a)	Failure blocks flow of spray coolant to nozzles of spray header of train A of containment spray system, which reduces redundancy of spray system. No safety effect on system operation. Minimum containment spray requirements will be met by the flow of containment spray coolant from the operation of train B (train A).	Failure reduces the redundancy of providing coolant spray to the containment. Fluid flow from CS pump 1 (pump 2) will be lost. Minimum flow requirements for containment spray will be met by CS pump 2 (pump 1) delivering working fluid to spray header in train B (train A).
CS Operation Phase	Containment spray injection and recirculation phases	Containment spray injection and recirculation phases
Failure Mode	Fails to open on demand	Fails to deliver working fluid
Component	1. Motor-operated gate valve 1-9001A (1-9001B analogous)	2. Containment spray pump 1 (pump 2 analogous)

TABLE 6.2.2-5 (SHEET 2 OF 2)

Remarks	Valves are opened by operator during the switchover from the RWST to the containment sumps (paragraph 6.2.2.2).	Valves are closed by the operator during the switchover from the RWST to the containment sumps (paragraph 6.2.2.2).
Failure <u>Detection Method^(b)</u>	Valve position indication (closed to open position change) at CB. Monitor light and alarm (valve open) for group monitoring of components at CB.	Valve position indication) open to closed position change) at CB. Monitor light and alarm (valve closed) for group monitoring of components at CB.
Effect on System Operation ^(a)	Failure blocks flow of coolant from the containment sump to the suction of CS pump 1 (pump 2). Coolant flow from CS pump 1 (pump 2) will be lost which reduces the redundancy of spray system. No safety effect on system operation. Minimum flow requirements for containment spray will be met by CS train B (train A).	Failure reduces the redundancy of isolation provided to prevent coolant flow from containment sump to RWST. No safety effect on system operation. Check valve 001 (008) provides isolation.
CS Operation Phase	Containment spray recirculation phase	Containment spray recirculation phase
Failure Mode	Fails to open on demand	Fails to close on demand
Component	Motor-operated gate valve 1-9002A or 1-9003B analogous)	. Motor-operated gate valve 1-9017A (1- 9017B analogous)
	က်	4.

5. Deleted.

a. List of abbreviations and acronyms:

CB - Control board
CI - Containment isolation
CS - Containment spray
RWST - Refueling water storage tank

b. As part of plant operation, periodic tests, surveillance inspections, and instrument calibrations are made to monitor equipment and performance. Failure may be detected during such monitoring of equipment, in addition to detection methods noted.

TABLE 6.2.4-1 (SHEET 1 OF 17)

CONTAINMENT PENETRATION/ISOLATION VALVE INFORMATION

		Normal Direction of Flow	Ont																							Ont																				
	Power		< 0	> د	x 2	¥ 2	Æ:	¥.	¥:	Y.	B (dc)	¥:	¥	¥ ×	ξ α	2 2	۲ <u>۹</u>		Y N	¥	¥	¥	<	(⋖	< 0	ם מ	۵Ž	¥	ΑN	ΑA	Y.	A (dc)	۲ <u>۲</u>	ž	ΑN	Ϋ́	¥ ź	۷ ×	<u> </u>		Ϋ́	ΑN	ΑN	<	<
	Valve	Closure Time (s)	5(0)(1)	_ ≤	≨ ≤	≨ ≤	≰:	≰:	≰:	≰:	≰:	≰:	≨ ≤	≨ ≤	\$	_ ≤	⊊ ⊴		. ₽	. ≼	. ≼	¥	4	Ş	≸	5(0)(r)	_ ≤	⊊ ≤	: ≰	≰	⋠	≰:	≤ ≤	≨ ≤	. ≤	¥	≰:	≰ :	≤ ≤	S		. ≰	≰	¥	≰	≨
	,		LO L																					_	_	ıΩ																			_	2
		Actuation Signal	II.	Dranga (Signal	o keepool.	≨ ≤	≰:	≰:	≰:	.	Remote m	≰:	≨ ≤	≨ ≤	\$,	⊒ ≤	⊊ ⊴	; ;	į	. ≼	. ≼	¥	-	į	S⊓	II S). 	PIOCESS SIGNAL	. ≼	≰	⋠	≰.	kemote m	≨ ≤	. ≤	¥	≰:	≰ :	≤ ≤	5	: <u>;</u>	. ≼	≰	¥	II.S	∏S
		Power /	O (,	0, Q																			¥		0, Q
		Post- Accident																				_		-	_					_	_					_	_					_	_	_		
	Valve Position		00) (ی ر	ی د) ن	· د	0	0	0	00	ט כ	ی ر) C) C) C) C	0 0	0	0	O	()	O	00	ی ر	טט	0	O	O	0) C	ى د	0	O	O	00) C	ט כ	0 0	0	O	O	O	O
	Valve		0 () د	ی ر	ه د	ه د	ا ن	O (0	0	ပ (ט כ	ی ر	ى د) د) C) C	C	0	0	O	C)	O	00	ی ر	טט	O	O	O	0) (ى د	0	O	O	00	၁ (ی د	C	O	O	O	O	O
		Normal	00) C	ی ر	ه د	ه د	ا ن	O (0	0	ပ (ט כ	ی ر) C) C) C	o C	C	C	C	O	C)	0	0 0) C	ט כ	0	O	O	0	o (ى د	0	O	O	00	ပ	ى د	00	O	O	O	0	0
_		lany	Remote man.	mon.	e man.										Pemote man	i alli		man								e man.	e man.													nem	Semote man.					
	Actuation Mode	Secondary	Remote	Demote man	Kemor	None	None:	None	None		None				Pomot	None	Non	Remote man	None	None	None	None	None	2	None	Remote man	Remote man.	None	None	None	None	None	None	None	None	None	None	None	None	Remote man	Remote	None	None	None	None	None
	Actual	ary									ote man.	Manual	<u> </u>	- Tal	B	-	<u> </u>	3	<u>-</u>	- E	er	lar											Kemote man.	<u> </u>	<u> </u>	ler	lar	la i	<u> </u>	2		lal	ral	lal		
;		al Primary	Auto	Auto	Auto	Auto	Anto	Anto	Anto	Anto	Rem:	Man	Man	Manual	Airbo	Maria	Manual	Airb	Manual	Manual	Manual	Manual	Airb	2	Auto	Auto	Auto	Auto	Auto	Anto	Anto	Anto	Kem	Manual	Manual	Manual	Manual	Manual	Manual	Airb	Auto	Manual	Manual	Manual	Auto	Auto
j		Essential or Nonessential																																												
			E/H(o), SMA(o) N	2 4	υ2	Z 2	z:	Z	Z:	Z	Totor E	z :	z	Z 2	z z	2 2	2 2	. 2	: z	: z	z	z	2	2	Z D	E/H(o), SMA(p) N	Zμ	υZ	z	z	z				. z	z _	z _	z 2	z	2 2	z	. z	z _	z _	Z	Z P
	Valve	Operator	, E		<u> </u>	o dell	Sell Sell	Se#	Sel	Self	Elec. motor	Manua	Manual	Manual	Air	Mania	Manual	Air	Mania	Manual	Manual	Manual	Solonoid	2	Solenoid	ОН ОН ОН ОН ОН ОН ОН ОН ОН ОН ОН ОН ОН	5	S G	Seff	Self	Self	Self	Elec. motor	Manua	Manual	Manual	Manual	Manual	Manual	Mariual	Ā	Manual	Manual	Manual	Solenoid	Solenoid
		Туре	Gate	Cate	Globe	Kelet	Yelet S	Kelet	Relet	Relief	Gate	Globe	Cate	Clope	ologo olog ologo olog ologo ologo ologo ologo ologo ologo ologo ologo ologo ologo ologo olog ologo olog ol	Globe	9000	Glob edole	Globe	Globe	Globe	Globe	Solonolog	2000	Solenoid	Gate		Relief	Relief	Relief	Relief	Relief	Gate	Globe	Globe	Globe	Globe	Globe	Globe	e dolo	Globe	Globe	Globe	Globe	Solenoid	Solenoid
	Length	of Pipe (ft-in.)	20'-9"	45.40"	01-00	8-07	-6-0Z	.50B.	20'-9"	209"	10-1		-DL-/	28-0	42,4	12.50	30"-10"		10,-01	45'-0"	! ,					28'-49/16"	30-11 9/10	27.5	279"	27'-9"	27'-9"	27'-9"	-BG	1.7"	14'-1"	27'-0"	26'-11"	31'-11"	40-6	21.0	57'-10"	46'-5"	26'-0"			
		Type Tests	∢																							<																				
	Location Relative to Con-	ainment inside/ Outside	= 1	= +	= 1	= 1	Ψ.	=	¥ .	¥	¥		= 1	= 1	= =	= =	= =	:=	:=	: =		ont	t	=	¥	= 1	= 1	==	: =	¥	щ	¥	¥		. =	¥	¥	= 1	= 1	= =	< ==	. =	¥		¥	¥
	725	重正〇	ō ō	5 6								⊆ (5 6										0 1/4	5	AV3 Out	V A		ō ō											5 6						M/1 Out	AV3 Out
		Valve Number	HV-3006A	2000	2000-700	PSV-3001	PSV-300	PSV-3003	PSV-3004	PSV-3005	HV-3009	176	088	330	A-132 HV-1300	× 447	X-119	HV-1300	X-200	X-211(0)	V 420	X-438(p)	HV3006AM/1	(s)(d)	HY3006MV3 (p)(s)	HV-3016	HV-3016	PSV-3011	PSV-301	PSV-3013	PSV-301	PSV-301	HV-3019	080	X-196	X-207	X-162 ^(U)	X-121	X-123	358	HV-13007R	X-215	X-166(P)	X-430	HY3016MV1 (p)(s)	HY3016MV3 (p)(s)
	Valve	Arrange- ment Fig. 6.2.4-1																																												
	>	₹ E Ø	9-3	2																						9.3	5-5																			
		Drawing Number	1X4DB159-3	244001																						1X4DB159-3	2X4DB10																			
		ESF or Support Systems	Yes																							Yes																				
		Line Size (in.)	29.5	0.02	p 0	٥٥	، و	9	9	9	4	. 2	4 4	720	0.70			- 4			· -	_	u	ś	-	29.5	G. S.	0 (5	9	9	9	9	4.0	v <	0.75	_	_			- 4	. 4	-	_	_	5.	_
			dary	=																						dary	¥																			
		Fluid	am Second	000																						am Secor	coolar																			
		me	n and stea	- Door	b arriver																					n and ste	Teed-	ia Mei																		
		System Name	Main steam and steam Secondary	to auxiliary recu-	water pur																					Main steam and steam Secondary	to auxiliary reed-	water puri																		
		GDC or RG	22(1)		-																					22(1)																				
		Penetra- tion Number																																												
		± ≠ ∠	-																							2																				

TABLE 6.2.4-1 (SHEET 2 OF 17)

	Normal Direction of Flow	Oort	no		Out	Out
Power	1E Bus A or B	<pre><pre><pre><pre><pre><pre><pre><pre></pre></pre></pre></pre></pre></pre></pre></pre>	4m4ZZZZZZZZZZZZZZZ	₹ ₹	N N N N N	ZZZZZ ZZZZŻ
Valve	Closure Time (s)	\$\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\	\$\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\	≨≨	\$\$\$\$	₽₹₹₹₹ \$
	E 1	SLU SLU SLU SLU SLU SLU SLU SLU SLU SLU	√Signal			
	Actuation Signal	28 29 27 24 24 24 24 24 24 24 24 24 24 24 24 24	S SL Process Signal NA A NA	≨≨	N N N N	\$\$\$\$\$\$
ı	Power	CCC44444444446644666466666666666666666	CCC44444444444444444444444444444444444	≨≸	₽₹₹ ₹	
ition	Post- Accident	000000000000000000000000000000000000000	000000000000000000000000000000000000000	υυ	0000	000000
Valve Position	Shutdown	000000000000000000000000000000000000000	000000000000000000000000000000000000000	00	0000	000000
	Normal	000000000000000000000000000000000000000	000000000000000000000000000000000000000	OO	0000	000000
	K	ran. ran. ran.	ran. ran. ran.		nan.	nan.
Actuation Mode	Secondary	Remote man. Remote man. Remote man. None eman. None Mone None None None None None None None None None None None None None None None None None None	R Remote man. Remote man. Remote man. None eman. None None None None None None None None None	None	Remote man None None None	Remote man. None None None None
Actuatic	ΔĬ			B	व व व	व व व व
	al Primary	Auto Auto Auto Auto Auto Auto Auto Auto	Auto Auto Auto Auto Auto Auto Auto Auto	Manual NA	Auto Manual Manual Manual	Auto Manual Manual Manual Manual
	Essential or Nonessential	ZZWZZZZZZZZZZZZZZZ	ZZWZZZZZZZZZZZZZZZ Z Z			zzzzz
		MA(t)	MA(p)	<u>8</u>	<u></u>	
Valve	Operator	EHHO, ST EHHO, ST EHHO, ST EEH SOEF SOEF SOEF SOEF SOEF SOEF SOEF SOEF		Manual NA	Air Manual Manual manual	Air Manual Manual Manual Manual
	Туре	Gate Gode Relief Relief Relief Relief Gode Gode Gode Gode Gode Gode Gode Gode	Gate Gate Gate Gate Gate Gate Gate Gate	Globe Flange	Globe Globe Globe	Globe Globe Globe Globe
Length	of Pipe (ff-in.)	25.14 3.44 1.65.14 3.44 1.65.14 3.44 2.66.19 3.65 2.66.19 3.65 3.95.10 3.95 5.05.10 5.65 5.05.10 5.65 5.05 5.05 5.05 5.05	20.07 20.17 20.09 34 20.09 34 20.09 34 20.09 34 27.10		10,	1.0"
	Type Tests	<	∢	ш	<	∢
Location Relative to Con-	tainment Inside/ Outside	70000000000000000000000000000000000000	00000000000000000000000000000000000000	n Out	ğeee	oeeeee
	Valve	HV3028A HV3028B PV-3028 PSV-3021 PSV-3022 PSV-3023 PSV-3023 PSV-3025 PSV-3025 PSV-3025 HV3008A	HV-3038A PV-3038B PV-3030 PSV-3031 PSV-	X-017	HV-7603A 126 335 409	HV-7603B 129 336 X-157 X-164 410
Valve	Arrange- ment Fig. 6.2.4-1	Ξ	-	56 ^(k)	34	46
		1X4DB156-1 2X4DB156-1	1X4DB159-1 2X4DB159-1	1X4DB159-1 2X4DB159-1	1X4DB159-3 2X4DB159-3	1X4DB159-3 2X4DB159-3
	t Drawing	1X4D1 2X4D1	2X4DD	1X4D	1X4DI 2X4DI	1X4DI 2X4DI
	ESF or Support Systems	Yes	Yes	9	°Z	°Z
	Size (n.)	8,8,8,8,8,8,9,9,9,9,9,9,9,9,9,9,9,9,9,9	82888888888888888888888888888888888888	0.75	3 1.5 0.75	3 1.5 0.75 1
	Fluid	Secondary	Secondary	N/A	Secondary	Secondary
	System Name	Main steam line	Main steam line	Eddy current/sludge lancing	Steam generator blowdown	Steam generator blowdown
	stra- GDC ber or RG	24(1)	927(1)	A/N	57(1)	₅₇ (i)
	Penetra- tion Number	m	4	c)	^	œ

TABLE 6.2.4-1 (SHEET 3 OF 17)

le mon	Direction of Flow	Ont	Out	드	Ont	Ont	드	Ont	Out
Power Source	Bus A	[®] .44444	# \$ \$ \$ \$	\$ \$ \$	₹ ₹₹₹	ω \$ \$ \$ \$	₹₹₹	m \$ \$ \$ \$ \$ \$ \$	₹ ₹₹₹
Valve	Time (s)	N N A A N	NA NA A	4 4 4 2 2 2	NA N	N N N N N N N N N N N N N N N N N N N	A A A	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	NA NA S
				Remote man. (h) N NA NA	ote man. ote man.	ote man. ote man.	Remote man. (h) N NA NA	ote man. ote man.	ote man.
	er Actuation re Signal	A A A A A	A S S S	Rem NA M	AFS Rem NA MA	AFS Rem NA NA	Z Z Z Z Z Z	AFS Rem NA NA NA NA	AFS Rem NA NA
ı	Power t Failure	N Z Z Z Z O Z Z Z Z	TZZZZ	DAA	CCCAA	00044 00044	NAN	CCCAAAA	CCCAA
Valve Position	Post- wn Accident	00000	0000	υ ^Ω υ	00000	00000	ο ^Ω ο	000000	00000
Valve F	Shutdown	00000	0000	090	00000	00000	090	0000000	00000
	Norma	00000	0000	090	00000	00000	090	0000000	00000
Actuation Mode	Secondary	Remote man. None None None	Remote man. None None	None None None	Remote man. None None None	Remote man. None None None	None None None	Remote man. None None None None None	Remote man. None None None
Actual	Primary	Auto Manual Manual Manual	Auto Manual Manual Manual	Remote man. Manual Manual	Auto Remote man. Remote man. Manual	Auto Remote man. Remote man. Manual	Remote man. Manual Manual	Auto Remote man. Remote man. Manual Manual	Auto Remote man. Remote man. Manual
	Essential or Nonessential	ZZZZZ	zzzz	zzz	zzzzz	zzzzz	zzz	zzzzzz	zzzzz
	ator	a a a a	a a a	la la	ooid ooid aal	ooid ooid aal	<u>n</u> n	ooid ooid aal aal	ooid noid al
Valve	Operator	Air Manual Manual Manual	Air Manual Manual Manual	Air Manual Manual	Solenoid Solenoid Solenoid Manual Manual	Solenoid Solenoid Solenoid Manual Manual	Air Manual Manual	Solenoid Solenoid Solenoid Manual Manual Manual	Solenoid Solenoid Solenoid Manual Manual
	Type	Globe Globe Globe Globe	Globe Globe Globe	Globe Globe Globe	Globe Globe Globe Globe	Globe Globe Globe Globe	Globe Globe Globe	Globe Globe Globe Globe Globe Globe	Globe Globe Globe Globe
Length	(ft-in.)	1'-0"	1'-0"	2'-0"	2'-6"	2'-6"	2'-0"	3. 2. 6.	2'-6"
	Type Tests	∢	∢	O	∢	∢	O	∢	∢
Location Relative to Con-	Inside/ Outside	ğ E E E E E E E	o = = =	n n Out	9 0	0 = = 0 =	o e e	9 6 6 7 6 7 7	- O O
	Valve	HV-7603C 132 337 X-155 407	HV-7603D 135 338 408	HV-5280 676 081	HV-9451 HV-9553A HV-9553B 047 043	HV-9452 HV-9554A HV-9554B 048	HV-5281 677 084	HV-9453 HV-9555A HV-9555B 049 045 X-185 X-423(o)	HV-9454 HV-9556A HV-9556B 050 046
Valve Arrange-	ment Fig. 6.2.4-1	34	34	3A	35	35	3A	32	35
	Drawing Number	1X4DB159-1 2X4DB159-1	1X4DB159-1 2X4DB159-1	1X4DB159-1 2X4DB159-1	1X4DB159-3 2X4DB159-3	1X4DB159-3 2X4DB159-3	1X4DB159-1 2X4DB159-1	1X4DB159-1 2X4DB159-1	1X4DB159-1 2X4DB159-1
G T	(0)	No 2	No 2	No 2	No 2	No 2	No 1- 2	N 0 1	2 1
<u>.</u>	Size (in.)	3 1.5 0.75	3 1.5 0.75	0.5	0.5 0.5 0.5 0.5 0.375	0.5 0.5 0.5 0.5 0.375	0.5 0.5 0.5	0.5 0.5 0.5 0.5 0.375 0.5	0.5 0.5 0.5 0.5 0.375
	Fluid	Secondary	Secondary	Secondary coolant chemicals	Secondary	Secondary	Secondary coolant chemicals	Secondary	Secondary
	System Name	Steam generator blowdown	Steam generator blowdown	Chemical addition	Steam generator secondary side sample	Steam generator secondary side sample	Chemical addition	Steam generator secondary side sample	Steam generator secondary side sample
	or RG	92(1)	57(1)	54	57(1)	57(1)	54	924(1)	57 _(i)
Penetra		ō	10	11A	118	110	12A	12B	12C

TABLE 6.2.4-1 (SHEET 4 OF 17)

	Normal Direction of Flow	Ont	E					드	£	<u>=</u>	٤	드	<u>=</u>
Power	1E Bus A or B	e a ₹	Y y B					A A	A,B	A A A	N A B	A A A	4 4 4 4 4 2 2 2 2 2
Valve	sure												
N	응투(s)	₹ ₹	t t ₹	'	'	•	•	₹ ₹	ς ¥	υ X X	φŽ	υ X X	\$ \$ \$ \$\$
	Actuation Signal												
		\$85€	\$85	•	•	•		≨≨	≖₹	π₹≸	πŹ	π₹₹	\$ \$ \$ \$\$
İ	Power T Failure	55₹	55₹		•			\$ \$	2 ₹	2₹₹	₽₹	2₹₹	\$\$\$\$ \$
sition	Post- n Accident	000	000					99	00	000	00	000	9,099
Valve Position	Shutdown	000	000					(c) (c)	00	000	υυ	000	9,099
	Normal	000	000					22	00	000	00	000	0,000
Actuation Mode	Secondary	Remote man. Remote man. None	Remote man. Remote man. None			1		None	Remote man. NA	Remote man. NA NA	Remote man. NA	Remote man. NA NA	None None None None
Ad	Primary	Auto Auto Manual	Auto Auto Manual					Manual Manual	Auto Manual	Auto Manual Manual	Auto Manual	Auto Manual Manual	Manual Auto Auto Manual Manual
	Essential or Nonessential												
1	Essen	zzz	zzz					zz	zz	zzz	zz	zzz	zzzzz
Valve	Operator	Solenoid Solenoid Manual	Solenoid Solenoid Manual					Manual Manual	E/H Manual	E/H Manual Manual	E/H Manual	E/H Manual Manaul	Manual Self Self Manual Manual
ļ	Type	Gate Gate Globe	Globe Globe Globe					Dia Dia	Gate Globe	Gate Globe Globe	Gate	Gate Globe Globe	Globe Check Relief Globe Globe
Length	of Pipe (ft-in.)	3,-0"	2'-0"					1-6	11'-0" 4'-1"	11'-0" 4'-5"	3,-0"	3-0"	2'-2"
	Type Tests	OO	00	<	<	<	<	O	<	<	∢	<	O
Location Relative to Con-	tainment Inside/ Outside	n O n	od = =					out E	Out	0 0 o	Ont	0 o o o	o e e e e
	Valve Number	HV-12975 HV-12976 X-001	HV-12977 HV-12978 X-003	None	None	None	None	050	HV-5229 X-031	HV-5228 X-036(P) X-037(o)	HV-5230 X-073	HV-5227 X-075 ⁽⁰⁾ X-076 ^(P)	005 038 PSV-17589 X-065 X-950
Valve	Arrange- ment Fig. 6.2.4-1	36	45	48	24	24	54	37	12	12	12	12	38
_													
	Drawing Number	1X4DB213-2 2X4DB213-2	1X4DB213-2 2X4DB213-2	1X4DB131 2X4DB131	1X4DB113 2X4DB113	1X4DB113 2X4DB113	1X4DB113 2X4DB113	1X4DB130 2X4DB130	1X4DB168-3 2X4DB168-3	1X4DB168-3 2X4DB168-3	1X4DB168-3 2X4DB168-3	1X4DB168-3 2X4DB168-3	AX4DB190-2
	ESF or Support Systems	o Z	8	Yes	Yes	Yes	Yes	o Z	8	o Z	o Z	o Z	°Z
	Line Size (in.)			Tubing	Tubing	Tubing	Tubing	n n	16	9	16	1 1 1 6	2 2 1.0 1.0
		ment	ment							Δr	Ž,	Žį.	
	Fluid	Containment atmosphere	Containment atmosphere	DC 702 silicone oil	Water	Water	Water	Borated water	Secondary	Secondary	Secondary	Secondary	r Demin. water
	System Name	Containment air radioactivity monitor inlet	Containment air radioactivity moni- tor outlet	Containment pressure detector	Reactor vessel water level instrumentation	Reactor vessel water level instrumentation	Reactor vessel water level instrumentation	Purification water supply to refueling cavity	Feedwater	Feedwater	Feedwater	Feedwater	Demineralized water supply
	GDC or RG	26	26	1.141	1.141	1.141	1.141	54	_{57(i)}	57 ⁽ⁱ⁾	57(1)	57 ⁽ⁱ⁾	54
	Penetra- tion Number	13A	13B	13C	14A	14B	14C	15	18	19	80	21	22

TABLE 6.2.4-1 (SHEET 5 OF 17)

Normal Direction of Flow Actuation Signal NA NA NA CIA Power Failure NA NA FC FAI Post-Accident (£) 9,0 00 Valve Position Shutdown 9,000 Secondary
None
None
None
Remote mar Manual Manual Actuation Mode Remote man. Remote man. Elec. motor Elec. motor Self Operator Manual Self Manual Air Elec. mote Type Gate Check Globe Globe Globe B-fly B-fly Check B-fly B-fly 1-6 Type Tests C Valve Number 211 1184 226 HV-3502 HV-3548 Valve Arrange-ment Fig. 6.2.4-1 40 40 1X4DB138-2 1X4DB138-1 1X4DB138-2 2X4DB138-2 1X4DB138-2 1X4DB138-1 1X4DB140 2X4DB140 1X4DB121 2X4DB121 ESF or Support Systems ŝ 8 Size (in.) 1.5 1.5 0.75 0.5 10 10 0.75 Fluid Compressed air Water with corrosion inhibitors Water with corrosion inhibitors Borated water Hot leg sample System Name ACCW supply GDC or RG 54 Penetra-tion Number 23

TABLE 6.2.4-1 (SHEET 6 OF 17)

	Normal Direction of Flow	드			드	⊑	Ont	Ont	Ont
	1E Bus A or B	≪ ₪	§ §§	< m \$ \$ \$ \$ \$ \$ \$ \$	m≸≸≸	₹ ₹₹	ω	∢	ω
Valve	Closure Time (s)	Υ Υ Σ Σ	444 222	N N N N N N N N N N N N N N N N N N N	₹₹₹ 2222	4 4 4 4 2 2 2 2	¥ Z	₹ Ž	¥.
	Actuation Signal	<u> </u>	ĕ-ĕ VOZ	Remote man. CIA NA NA NA NA NA	SARA	NANA	Sl(e)	(e)	Remote man.
	Power Failure	₹₹	A D A	A S S S S S S S S S S S S S S S S S S S	ZZZZ ZZZZZ	Z Z Z Z	FAI	Æ	¥
sition	Post-	00	,00	(a) (o) (o) (o) (o) (o) (o) (o) (o) (o) (o)	0,00	0,00	(q) ^O	(q) ^O	(q)O
Valve Position	al Shutdown	υυ	,00	00,,00000	0,00	0,00	O	O	O
	Nomal	υo	,00	00,,00000	0,00	0,00	O	O	O
Actuation Mode	Secondary	Remote man. Remote man.	None Remote man. None	Manual Remote man. None None None None None	Remote man. None None	Remote man. None None	Remote man.	Remote man.	Manual
Actua	Primary	Auto	Auto Auto Manual	Remote man. Auto Auto Auto Manual Manual Manual Manual	Auto Auto Manual Manual	Auto Auto Manual Manual	Auto	Auto	Remote man. Manual
_	Essential or Nonessential	шш	шZZ	wzwwzzzzz	шшzz	шшzz	ш	ш	ш
Valve	Operator	Elec. motor Elec. motor	Self Air Manual	Elec. motor Air Self Self Manual Manual Manual Manual	Elec. motor Self Manual Manual	Elec. motor Self Manual Manual	Elec. motor	Elec. motor	Elec. motor
	Type	Gate Gate	Check Globe Globe	Gate Globe Check Check Globe Globe Globe Globe	Gate Check Globe Globe	Gate Check Globe Globe	Gate	Gate	Gate
Length	of Pipe (ft-in.)	16'-5"			2-0" 2-0" 2-0"	7-5" - 2'-1" 2'-1"	99	99	.69
	Type Tests	∢		∢	O	O	∢	<	∢
Location Relative to Con-	tainment Inside/ Outside	om to	드드드	Q E E E E E E E E	out out	out out out	Out	Ont	Ont
	Valve	HV-8801A HV-8801B	013 HV-8843 173	HV-8802A ⁽¹⁾ HV-8881 120 121 290 X-292 X-293 X-294 X-295	HV-9001B 016 014 046	HV-9001A 015 013 047	HV-8811B ^(I)	HV-8811A ^(I)	HV-9002B ^(I)
Valve	Arrange- ment Fig. 6.2.4-1	25		20	56	56	27	27	23
	Drawing Number	1X4DB119 2X4DB119		2X4DB121	1X4DB131 2X4DB131	1X4DB131 2X4DB131	1X4DB122 2X4DB122	1X4DB122 2X4DB122	1X4DB131 2X4DB131
	ESF or Support Di Systems Ni	Yes 1)		Yes 1)	Yes 1)	Yes 1)	Yes 1)	Yes 1)	Yes 1)
	Size (in.)	44	3 0.75 0.75	7 0.75 0.75 0.75 0.75 0.75 0.75 0.75 0.7	8 8 2 0.75	8 8 2 0.75	4	4	12
	Fluid	Borated water		Borated water	Borated water	Borated water	Borated water	Borated water	Borated water
	System Name	Boron injection line to cold leg		Safety injection to hot leg	Containment spray supply	Containment spray supply	RHR emergency sump suction	RHR emergency sump suction	Containment spray emergency sump suction
	GDC or RG	55		55	99	99	26	99	26
	Penetra- tion Number	32		33	34	35	36	37	38

TABLE 6.2.4-1 (SHEET 7 OF 17)

	Normal Direction of Flow	Ont	드	o ort	드	<u>=</u>
Power	1E Bus A or B	<	N A A	4 m m 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	4 Z Z	4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4
Valve	Closure Time (s)	≨	844		£ ≹ ₹	922222222222222
>	o⊨의		X Z Z		₩ 2 2	¥ZZZZZZZZZZZZZZZZ
	Actuation Signal	Remote man.	ĕ₹ĕ	<u> </u>	SES	<u></u>
	Power	FA	2₹₹	CCC33333333	₽₹≸	<u>₹</u> ₹₹₹₹₹₹₹₹₹₹₹₹
osition	Post-	(g) ^O	0,0	000000000	0,0	0000000000000000
Valve Position	Shutdow	O	0.0	0000000000	0,0	0000000000000000
	Normal Shutdown	O	0,0	000000000	0,0	0000000000000000
Actuation Mode	Secondary	Manual	Remote man. None None	Remote man. Remote man. Remote man. None None None None None None	Remote man. None None	Remode man. None None None None None None None None
Actua	Primary	Remote man.	Auto Auto Manual	Auto Auto Auto Manual Manual Manual Manual Manual	Auto Auto Manual	Auto Auto Auto Auto Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual
	Essential or Nonessential					
ı	Esser	ш	zzz	zzzzzzzzz	zzz	ZZZZZZZZZZZZZZZ
Valve	Operator	Elec. motor	Air Self Manual	Air Air Air Manual Manual Manual Manual Manual	Air Self Manual	Elec. motor Seff Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual
	Type	Gate	Gate Check Gate	Globe Globe Globe Globe Globe Globe Globe Globe	Globe Check Globe	Relef (1) (2) (3) (4) (4) (4) (4) (4) (4) (4) (4) (4) (4
Length	of Pipe (ft-in.)	69	2'-6"	37" 22-9" 22-9" 22-6" 6-8" 11-6" 11-0" 33-7"	2'-6"	3-0-
	Type Tests	∢	O	O	O	∢
Location Relative to Con-	tainment Inside/ Outside	Ont	n n O	- T T T T T T T T T T T T T T T T T T T	our i	ğ
	Valve	HV-9002A ^(I)	HV-27901 036 018	HV-8964 HV-8964 HV-8988 016 X-465 X-444(o) X-835(o) 293(b) 324(p) PSV-8871	HV-8880 017 013	HV-2134 PSV-11673 255 X-668 X-185 X-186 X-181 N-182(p) 309 317 323 325 326 346 348 348
Valve	Arrange- ment Fig. 6.2.4-1	23	14		m	71
	Drawing r Number	1X4DB131 2X4DB131	1X4DB174-4 2X4DB174-4	1X4DB121 2X4DB121	1X4DB120 2X4DB120	2X4DB135-1 2X4DB135-1
		××	X X	* %	¥ %	**
	ESF or Support Systems	Yes	Š	°Z	Š	° Z
	Line Size (in.)	12	49←	0.75 0.75 0.75 0.75 0.75 0.75 0.75 0.75	1 1 0.75	8 0.75 0.55 1.11 1.11 1.12 1.13 1.13 1.13 1.13 1.13
	Fluid	Borated	Well	Borated	ž	Treated well
	System Name	Containment spray emergency sump suction	Fire protection water	Accumulator test and drain line	Nitrogen supply to accumulator	NSCW supply to reactor cavity codes
	or RG	26	54	54	54	57
	Penetra- tion Number	99	9	14	42	64

TABLE 6.2.4-1 (SHEET 8 OF 17)

	Normal Direction of Flow	Ort	ے	pno		
	1E Bus A or B	45555555555555555555555555555555555555	m24544444444444444444444444444444444444	m \$5\$		
Valve	Closure Time (s)	9222222222222	92222222222222222	\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$		
	Power Actuation Failure Signal	\$\frac{1}{2}\frac{1}{2	######################################	52222222222222222222222222222222222222		
Valve Position	Post- down Accident	000000000000000	000000000000000000000000000000000000000	000000000000000000000000000000000000000		
Valve	Normal Shutdown	000000000000000	000000000000000000000000000000000000000	0000000000000000000		
Actuation Mode	Secondary	None of the control o	N N N N N N N N N N N N N N N N N N N	R Remote Tanna R Remote Tanna R Remote Tanna R Remote Tanna R Remote Tanna R Remote Tanna R R Remote Tanna R R R R R R R R R R R R R R R R R R		
Ad	Primary	Auto Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual	Auto Nanua Nanua Nanua Nanua Nanua Nanua Nanua Nanua Nanua Nanua Nanua Nanua Nanua	Auto Nanua		
	Essential or Nonessential	zzzzzzzzzzzz	zzzzzzzzzzzzzzzzzzzzz	zzzzzzzzzzzzzzzzzzzzzzzzzzzzzzzzzzzzzz		
Valve	Operator	Elec. motor Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual	Elec. motor Nama al Manual	Efec. motor Manual		
	Type	P-ff G100be G100be G100be G100be G100be G100be G100be G100be	- 0 0	# 9 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0		
Length	of H-in.)	2.0"	0,000 0,000	- 1 - 1 - 6 - 1 - 6 - 1 - 1 - 6 - 1 - 1		
	Type Tests	∢	∢	∢		
Location Relative to Con-	tainment Inside/ Outside	ğ==========	222222222222000	22222222222222000		
	. Valve Number	HV2138 X-183 X-184 (P) X-187 X-186 138 139 131 311 311 311 312 327 327 350 352 352	H V 2135 H V 2135 H V 2137 H V 21	HV.2139 HV.2139 1040 1041		
Valve	Arrange- ment Fig. 6.2.4-1	94	4	94		
	t Drawing s Number	1X4DB135-1 2X4DB135-1	1X4DB135-2 2X4DB135-2	1X4D B135-2 2X4D B135-2 2X4D B135-2		
	Support Systems	° ž	o z	° Z		
	Size (ii.)	8 1 0 0.5 0 0.75 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	8 0.75 0.75 0.05 0.05 0.05 0.05 0.05	8		
	Fluid	Treated well water	Treated well water	Treated well water		
	System Name	NSCW return from reador cavity coolers	Coders cavity coders	NSCW return from reactor cavity coders		
	- GDC	22	23	24		
	Penetra- tion Number	4	8	94		

TABLE 6.2.4-1 (SHEET 9 OF 17)

	Normal Direction of Flow	Ont	Ont	드		드	드	드	드	, ₹	드	드	E
Power	1E Bus A or B	∢ω≸	m∢≸	В	≨≸	m₹≸	m₹≸	m≸≸	m₹≸	≨	∞ ₹ ₹ ∞ ₹ ₹ ₹	a ₹ ₹ § ®	ω ₹ ₹ ₹ œ ₹
Valve	Closure Time (s)	75 TS	₹ £ ₹	17(m)	₹₹	\$\$\$	\$\$\$	\$\$\$	\$\$\$	₹₹	\$ \$ \$ ¢ ¢ \$ \$ \$	\$ \$ \$ \$ \$ \$ \$	\$ \$ \$ \$ \$
	- Actuation Signal	¥ ĕ ĕ	¥ oo s	S	₹ ₹	Remote man. NA NA	Remote man. NA NA	Remote man. NA NA	Remote man. NA NA	\$ \$	Remote man. NA NA CIA NA NA NA	Remote man. NA NA NA CIA	Remote man. NA NA NA CIA CIA
1	Power Failure	요요≹	₹₹≸	FA	≨≨	<u>₹</u> \$\$	₹ ≸≸	₹₹₹	₹ ₹ ≸	≨≸	<u> </u>	<u>₹</u> ₹₹\$0	<u>₹</u> ₹₹₹2₹
osition	Post- wn Accident	000	00,	O	٠ ٥	0,0	0,0	0,0	0,0	υυ	(a) 0	(g) O O O	(a) 0 0 0 0
Valve Position	Shutdown	000	00,	0	, 0	0,0	0,0	0,0	0,0	00	0,,0000	0,,00	0,,000
	Normal	000	00.	0	۰, ٥	0,0	0,0	0,0	0,0	υυ	0 , , 0 0 0 0	0 , , 00	0 , ,000
Actuation Mode	Secondary	Remote man. Remote man. None	Remote man. Remote man. None	Remote man.	None None	Manual None None	Manual None None	Manual None None	Manual None None	None None	Manual None None Remote man. None None	Manual None None None Remote man.	Manual None None None Remote man.
Actual	Primary	Auto Auto Manual	Auto Auto Auto	Auto	Auto Manual	Remote man. Auto Manual	Remote man. Auto Manual	Remote man. Auto Manual	Remote man. Auto Manual	Manual	Remote man. Auto Auto Auto Manual Manual	Remote man. Auto Auto Manual Auto	Remote man. Auto Auto Manual Auto Manual
	Essential or Nonessential	zzz	zzz	z	zz	шшZ	шшz	шшz	шшZ	zz	www.zzzz	wwwzz	www.zzz
/alve	Operator	Air Air Manual	Elec. motor Elec. motor Self	Elec. motor	Self Manual	Elec. motor Setf Manual	Elec. motor Self Manual	Elec. motor Self Manual	Elec. motor Self Manual	Manual NA	Elec. motor Self Self Air Manual Manual	Elec. Motor Self Self Manual	Elec. motor Self Self Manual Air
	Type	Globe Globe	Globe Globe Check	Gate	Check Globe	Globe Check Globe	Globe Check Globe	Globe Check Globe	Globe Check Globe	Globe ^(q) Flange	Gate Check Check Globe Globe Globe	Gate Check Check Globe Globe	Gate Check Check Globe Globe
Length	of Pipe (ff-in.)	4'-5"	2'-4"	3,-0		1-3"	1-3"	±+	<u>-</u>		3,-0"	2,-0"	1-10"
	Type Tests	O	O	O		∢	∢	∢	∢	Ф	⋖	∢	∢
Location Relative to Con-	tainment Inside/ Outside	Out Out	한도도	Ont	드드	P P O	를 드 드 드 드 드	o e e	마마O	i Out	ğ = = = = = =	ğeeee	o e e e e e
	Valve	HV-8160 HV-8152 502(1)(5)	HV-8100 HV-8112 021	HV-8105	032 465	HV-8103D 355 452	HV-8103C 354 451	HV-8103B 353 450	HV-8103A 004 449	X-018	HV-8840(I) 128 129 HV-8825 112(O) X-435 296(P)	HV-8809A 147 148 111 HV-8890A	HV-8809B 149 150 110 HV-8890B X-410
Valve	Arrange- ment Fig. 6.2.4-1	_	31	33		32	32	32	32	56 ^(K)	21	30	30
	Drawing Number	1X4DB114 2X4DB114	1X4DB114 2X4DB114	1X4DB114		1X4DB114 2X4DB114	1X4DB114 2X4DB114	1X4DB114 2X4DB114	1X4DB114 2X4DB114	1X4DB159-1 2X4DB159-1	1X4DB121 2X4DB121	1X4DB121 2X4DB121	1X4DB121 2X4DB121
	ESF or Support Systems	8	8	ê		Yes	Yes	Yes	Yes	°Z	Yes	Yes	Yes
	Line Size (in.)	3 3 0.75	2 2 0.75	ဇ	3 0.75	1.5 1.5 0.75	1.5 1.5 0.75	1.5 1.5 0.75	1.5 1.5 0.75	0.75	12 8 8 0.75 0.75 0.75	8 6 6 0.75 0.75	8 6 6 0.75 1
	Fluid	Primary coolant	Primary coolant	Primary	5	o Primary coolant	o Primary coolant	o Primary coolant	o Primary coolant	∀ Y	Borated	Borated	Borated
	System Name	Normal letdown line	Excess letdown and seal water leakoff	Normal charging	2	Reactor coolant pump Primary seal water supply coolant (pump loop No. 4)	Reactor coolant pump Primary seal water supply coolant (pump loop No. 3)	Reactor coolant pump Primary seal water supply coolant (pump loop No. 2)	Reactor coolant pump seal water supply (pump loop No. 1)	Eddy current/sludge lancing	RHR pump discharge to hot leg	RHR loop into cold leg	RHR loop into cold leg
	GDC or RG	54	54	24		24	24	24	24	A A	55	55	55
	Penetra- tion Number	84	49	20		12	25	23	54	22	99	22	28

TABLE 6.2.4-1 (SHEET 10 OF 17)

. 0	Normal Direction of Flow	Out	Out	Out/In	드			Out	Out			드	드
	1E Bus A or B	₹Ž	в¥	m∢	a ž ž	Z Z	₹ ₹ Z Z	≪ ₪	⊠ ∢		<u> </u>	4 4 4 2 2 2	ZZZZ
Valve	Closure Time (s)	≨ ≸	≨ ≸	15 15	5 \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \	≨ ≨	≨ ≨	5 5	15		\$\$\$	₹ ₹₹	§ §§
	Actuation Signal	Remote man. NA	Remote man. NA	CIA	N N O	¥ ∀	¥ ¥	CIA	CIA		Z Z Z	Remote man. ^(h) NA NA	Remote man. ^(h) N NA NA
	Power	₹₹	₹₹	55		≨ ≨	≨ ≨	55	55		2 22	244	2₹₹
sition	Post- n Accident	υυ	υυ	OO	0,0	υŽ	υŠ	υυ	υυ		ožo	090	000
Valve Position	Shutdown	00	00	00	0 . 0	υ¥	o &	υυ	υυ		υŽυ	090	oco
	Normal	υυ	O	00	0.0	υ¥	o &	υυ	υυ		ONO	OCC	oΩo
Actuation Mode	Secondary	Manual None	Manual None	Remote man. Remote man.	Remote man. None None	None	None	Remote man. Remote man.	Remote man. Remote man.		None None	None None	None None
Actuat	Primary	Remote man. Auto	Remote man. Auto	Auto	Auto Auto Manual	Manual NA	Manual NA	Auto	Auto		Manual NA Manual	Remote Manual Manual	Remote Manual Manual
	Essential or Nonessential	шΖ	шZ	zz	zzz	zz	zz	zz	zz		zzz	zzz	zzz
Valve	Operator	Elec. motor Self	Elec. motor Self	Air Air	Air Seff Manual	Manual NA	Manual NA	Air	Air		Manual NA Manual	Air Manual Manual	Air Manual Manual
	Type	Gate Relief	Gate Relief	Dia Dia	Dia Check Globe	Globe Flange	Globe Flange	Globe	Globe		Globe Flange Globe	Globe Globe Globe	Globe Globe Globe
Length	of Pipe (ff-in.)			1-6"	1-6	,		4'-0"	3'-0"			1-3"	1-6"
	Type Tests	∢	∢	O	O	ω	ω	O	O	∢	ш	O	O
Location Relative to Con-	tainment Inside/ Outside	드드	드드	n Out	od = =	드드	드드	n Out	Ont		드드드	o e e	는 는 O
	Valve Number	HV-8701A PSV-8708A	HV-8702A PSV-8708B	HV-8033 HV-8047	HV-8028 112 020	119 NA	120 A	HV-3514 HV-3513	HV-3507 HV-3508		018 ^(U) NA 019 ^(P)	HV-5278 678 087	HV-5279 679 090
Valve	Arrange- ment Fig. 6.2.4-1	4	4	7	39	51	51	7	7	48	18	3B	3B
	Drawing Number	1X4DB122 2X4DB122	1X4DB122 2X4DB122	1X4DB112 2X4DB112	1X4DB112 2X4DB112	1X4DB132 2X4DB132	1X4DB132 2X4DB132	1X4DB140 2X4DB140	1X4DB140 2X4DB140	1X4DB131 2X4DB131	1X4DB132 2X4DB132	1X4DB159-3 2X4DB159-3	1X4DB159-3 2X4DB159-3
	ESF or Support Systems	Yes	Yes	o Z	° N	§.	§.	°Z	o N	Yes	o Z	o N	°Z
	Line Size (in.)	3 3	3 3		9.8	0.5	0.5	0.5	0.5	Tubing	0.75 8 0.75	0.5 0.5 0.5	0.5
	Fluid	Primary coolant	Primary coolant	Waste gas/N	Demin. water	Containment atmosphere	Containment atmosphere	Primary coolant	Primary coolant	DC 702 Silicone oil	Containment atmosphere	Secondary coolant chemicals	Secondary coolant chemicals
	System Name	RHR suction from hot leg	RHR suction from hot leg	Pressurizer relief tank sample to waste gas compressor suction and N supply	Pressurizer relief tank makeup water supply	Flow verification and pressure sensing piping	Flow verification and pressure sensing piping	Pressurizer steam sample line	Pressurizer liquid sample line	Containment pressure detector	Containment leak rate test	Chemical addition	Chemical addition
	GDC or RG	22	22	54	54	¥.	A A	55	22	1.141	₹ Z	54	54
	Penetra- tion Number	29	09	62	63	64A	64B	67A	67B	94C	88	99 V	969 869

TABLE 6.2.4-1 (SHEET 11 OF 17)

	Nomal Direction of Flow		Ont	드		Ont	드		Ont	Ont	Out	Out	Out
	1E Bus A or B		88	₹ ≸		484	m≱≸		₹ ₹	₹ ₹	m≸≸	ω≸≸	<m2555555555555555555555555555555555555< td=""></m2555555555555555555555555555555555555<>
Valve	Closure Time (s)	,	222	55 5	,	222	55 5	,	24 A	24 A	2 A A	\$ \$ \$	######################################
	Actuation Signal		Remote man. Remote man. Remote man.	Remote man. NA NA		Remote man. Remote man. Remote man.	Remote man. NA NA		NA A A	N N O	NA A A	NA A A	A A A A A A A A A A A A A A A A A A A
ı	Power Failure		555	SAA		555	SAA		SAA	SAA	SAA	DAA	CCAAAAAAA A
Valve Position	Post- An Accident		000	0,0		000	0,0		υ <u>9</u> υ	υgο	υ <u>9</u> υ	υ ^Q υ	0000000000000000
Valve F	Shutdown		000	0,0		000	0,0		090	090	090	ంలం	000000000000000
	Noma		000	0,0		000	0,0		090	090	090	ంకిం	00000000000
Actuation Mode	Secondary		Remote man. (n) None None	Remote man. None None		None Remote man. (n) None	Remote man. None Manual		Remote man. None None	Remote man. None None	Remote man. None None	Remote man. None None	Remote man. None None None None None None None None
Actu	Primary		Remote man. Remote man. Remote man.	Remote man. Auto Manual		Remote man. Remote man. Remote man.	Remote man. Auto Manual		Auto Manual Manual	Auto Manual Manual	Auto Manual Manual	Auto Manual Manual	Auto Auto Manual Manual Manual Manual Manual Manual Manual
,	Essential or Nonessential	,	шшш	шшz		шшш	шшz		zzz	zzz	zzz	zzz	zzzzzzzzz z
	Operator		Solenoid Solenoid Solenoid	Solenoid Self Manual		Solenoid Solenoid Solenoid	Solenoid Self Manual		Solenoid Manual Manual	Solenoid Manual Manual	Solenoid Manual Manual	Solenoid Manual Manual	Air Air Manual Manual Manual Manual Manual Manual Manual
Valve		•											
	Type		Globe Globe Globe	Globe Check Globe		Globe Globe Globe	Globe Check Globe		Globe Globe Globe	Globe Globe Globe	Globe Globe Globe	Globe Globe Globe	Dia Dia Globe Globe Globe Globe Globe Globe Globe Globe
Length			2-9"	1:-2"		1.9"	2-3		2'-9"	2'-4"	2'-9"	2'-4"	11.0"
	Type Tests	∢	O	O	∢	O	O	∢	O	O	O	O	O
Location Relative to Con-	tainment Inside/ Outside		o e e	o e e		n ou	o e e		n Q n	n Q n	n Q n	n Or	
	Valve Number	None	HV-2791A HV-2790A HV-2790B	HV-2793A 001 039	None	HV-2792B HV-2791B HV-2792A	HV-2793B 002 040	None	HV-10950 159 178	HV-10952 161 183	HV-10951 160 181	HV-10953 162 185	HV-7699 HV-7136 X-186(0) X-218(0) X-220(0) X-153(p) X-154(p) X-173(p) X-229(p) X-229(p)
Valve	Arrange- ment Fig. 6.2.4-1	48	4	60	48	4	00	48	28	28	28	28	59
	Drawing Number	1X4DB131 2X4DB131	1X4DB213-2 2X4DB213-2	1X4DB213-2 2X4DB213-2	1X4DB131 2X4DB131	1X4DB213-2 2X4DB213-2	1X4DB213-2 2X4DB213-2	1X4DB131 2X4DB131	1X4DB120 2X4DB120	1X4DB120 2X4DB120	1X4DB120 2X4DB120	1X4DB120 2X4DB120	1X4DB127 2X4DB127
	ESF or Support Systems	Yes	Yes	Yes	Yes	Yes	Yes	Yes	<u>8</u>	S S	o N	o Z	2
	Line Size (n.)	Tubing	0.75 0.75 0.75	0.75 0.75 0.75	Tubing	0.75 0.75 0.75	0.75 0.75 0.75	Tubing	0.75 0.75 0.75	0.75 0.75 0.75	0.75 0.75 0.75	0.75 0.75 0.75	3 3 1 1 1 1 1 1 0.75
				Containment atmosphere	Ē	Containment atmosphere							
	Fluid	re DC 702 silicone oil	Containment atmosphere	Contai	re DC 702 silicone oil	Contai	Containment atmosphere	re DC 702 silicone oil	Borated	Borated	Borated	Borated	n Primary s coolant
	System Name	Containment pressure detector	Containment H ₂ monitor suction	Containment H ₂ monitor discharge	Containment pressure DC 702 detector	Containment H ₂ monitor suction	Containment H ₂ monitor discharge	Containment pressure detector	Accumulator sample line	Accumulator sample line	Accumulator sample line	Accumulator sample line	Reactor coolant drain tank pump discharge
	or RG	1.141	26	26	1.141	26	26	1.141	54	54	54	24	45
	Penetra- tion Number	269	70A	70B	70C	417 A	718	71C	72A	72B	73A	73B	E

TABLE 6.2.4-1 (SHEET 12 OF 17)

	Normal Direction of Flow	out Out	Ont	<u>s</u>	드	E	Ont		E			
	1E Bus A D	A B A	< ω	- A A A A A A A A	- NA A NA A	- V M V M Š	× m × m ×		- m ∢ ₹	Y Y Y		
	Closure Time (s)	15 NA										
>	O 를 의	## # ≥	15	8888	₹ ¥ ¥	20 C S S S	¥ 2 2 1 0 1 0 1 0 1 0 1 0 1 0 1 0 1 0 1 0	'	€ 4 ₹	\$ \$\$	'	'
	Actuation Signal	Y Y Y	CIA	<u>4</u> 444	ĕ≸≸	\$5555₹	5555 4		A S S	\$ \$\$		
	Power A	202	55	D222	022	<u>₹</u> ₹222¥	<u>₹</u> ₹222¥		202	4 44		
	Post-											
Valve Position		000	υυ	0,00	0,0	00009	60000	'	000	υŽυ	'	•
Valve	al Shutdown	000	υυ	0,00	0,0	00009	00009		000	υ¥υ		
	Normal	000	00	0,00	0,0	99009	22009		000	oğo		
Aode	Secondary	Remote man. Remote man. None	Remote man. Remote man.	Remote man. None None None	Remote man. None None	Remote man. Remote man. Remote man. Remote man.	Remote man. Remote man. Remote man. Remote man.		Remote man. Remote man. None	Vone Vone		
Actuation Mode	δ)	žžž	άŭ	žžžž	žžž	~~~~~~	~~~~~	'	άŭž	žžž	'	'
	Primary	Auto Auto	Auto	Auto Auto Manual Manual	Auto Auto Manual	Auto Auto Auto Auto Manual	Auto Auto Auto Auto Manual		Auto Auto Manual	Manual NA Manual	,	
	Essential or Nonessential											
ĺ	Essel	zzz	zz	zzzz	zzz	ZZZZZ	ZZZZZ		zzz	zzz		
Valve	Operator	Air Self	Air	Air Seff Manual Manual	Air Self Manual	Elec. motor Elec. motor Air Air Manual	Elec. motor Elec. motor Air Air Manual		Solenoid Solenoid Manual	Manual NA Manual		
	Type	Gate Gate Relief	Dia Dia	Gate Check Globe Globe	Globe Check Globe	B-fly B-fly Gate	B-fly B-fly Gate		Globe Globe Globe	Globe Flange Globe		
Length	of Pipe (ff-in.)	3-9"	1-9"	2'-0"	1-6"	7'-0" 6'-0" 2'-3"	13'-0" 7'-0" 2'-3"		. 8-1-8			
	Type Tests	O	O	O	O	O	O	∢	O	m	∢	<
Location Relative to Con-	tainment Inside/ Outside	크 O 크	Out Out	n Ond	를 드 드 드 드	oor out	oor or out		n O ri	도도도		
	Valve Number	HV-780 HV-781 PSV-0780	HV-7126 HV-7150	HV-9385 034 228 229	HV-9378 049 256	HV-2626A HV-2627A HV-2626B HV-2627B 001	HV-2628A HV-2629A HV-2628B HV-2629B 001	None	HV-8211 HV-8212 X-002	019 ^(U) NA 018 ^(P)	None	None
Valve	Arrange- ment Fig. 6.2.4-1	10	6	ø	ω	15	47	48	45	18	54	54
		3143	3127	1X4DB186-1 2X4DB186-1	1X4DB186-4 2X4DB186-4	1X4DB213-1 2X4DB213-1	1X4DB213-1 2X4DB213-1	3131	3110	3132 3132	3113	3113
	t Drawing S Number	1X4DB143 2X4DB143	1X4DB127 2X4DB127	1X4DI 2X4DI	1X4D	1X4DI 2X4DI	1X4DI 2X4DI	1X4DB131 2X4DB131	1X4DB110 2X4DB110	1X4DB132 2X4DB132	1X4DB113 2X4DB113	1X4DB113 2X4DB113
	ESF or Support Systems	°Z	°Z	°Z	°Z	^o Z	^o Z	Yes	^o Z	°Z	Yes	Yes
	Line Size (in.)	3 3 0.75	0.75	4 4 0.75 0.75	2 2 0.75	24 24 14 14 0.75	24 24 14 14 0.75	Tubing	1.0	0.75 8.0 0.75	Tubing	Tubing
	Fluid	Drains	Gas	Compressed	Compressed	Containment atmosphere	Containment atmosphere	DC 702 silicone oil	Containment atmosphere	Containment atmosphere	Water	Water
	System Name	Normal containment sump pumps discharge	Reactor coolant drain tank vent and H ₂ supply	Service air and post- Compressed LOCA purge air supply air	Instrument air	Normal containment purge supply and equalizing	Normal containment purge exhaust and equalizing	Containment pres- sure detector	Post-accident sampling	Containment leak rate test	Reactor vessel water level instrumentation	Reactor vessel water level instrumentation
	GDC or RG	54	54	26	54	26	26	1.141	26	₹ Z	1.141	1.141
	Penetra- tion Number	82	79	88	18	83	28	85C	86A	87	88A	88B

TABLE 6.2.4-1 (SHEET 13 OF 17)

	Normal Direction of Flow		,	,	٩	٩
Power	1E Bus A or B		₹ Z Z	¥ ¥	m ZZZZZZZZZZZZZZZZZZZZZZZZZZZZ	
Valve	Closure Time (s)		≨≨	\$ \$	\$ \$	\$ \$
	er Actuation		§ §	≨≨	~\$	<u></u>
	Power nt Failure		≨≸	≨≸	<u>₹</u> \$	<u> </u>
sition	Post-		oo	υ¥	000000000000000000000000000000000000000	000000000000000000000000000000000000000
Valve Position	Shutdown		© C	o ¥	000000000000000000000000000000000000000	000000000000000000000
	Normal		00	υ¥	000000000000000000000000000000000000000	00000000000000000000
Actuation Mode	Secondary		None None	None	Remote man. Note to the state of the state o	Remote man. Note to the control of t
Act	Primary		NA Manual	Manual	Auto Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual	Auto Manual Manual Auto Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual
	Essential or Nonessential		zz	zz	wzzzzzzzzzzzzzzzzzzzz	wzzzzzzzzzzzzzzzzzzz
Valve	Operator		NA Manual	Manual	Elec. motor Namual Manual	Elec. motor Manual
	Туре		Flange Globe	Globe Flange	P	P = 0.00
Length	of Pipe (ff-in.)				2 2.6.	7-1-6
	Type Tests	∢	ш	ш	∢	∢
Location Relative to Con-	tainment Inside/ Outside		드드	n Out	ğ	00
	. Valve Number	None	V 000	X-019	114/1809 019 114/1809 114/1805 114/1805 114/1809	100 HV-1807 100 HV-1815 100 H
Valve	Arrange- ment Fig. 6.2.4-1	54	13	56 ^(k)	<u>ي</u>	44
	Drawing Number	1X4DB113 2X4DB113	1X4DB130 2X4DB130	1X4DB159-1 2X4DB159-1	1X4DB135-2 2X4DB135-2	1X4DB135-2 2X4DB135-2
	ESF or Support Systems	Yes	§.	2	Yes	Yes
:	Size (in.)	Tubing	20	0.75	8 0.75 0.55 0.55 0.55 0.57 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	8 8 8 7 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	Fluid	Water	Refueling pool water	ν V	verial water water	Treated Verland
	System Name	Reactor vessel water level instrumentation	Transfer tube	Eddy current/sludge lancing	NSGW supply to containment coolers	NSGW supply to containment coolers
	GDC	1.141	₹ Z	₹	22	22
	Penetra- tion Number	88 SC	68	06	2	26

TABLE 6.2.4-1 (SHEET 14 OF 17)

	Normal Direction of Flow	=	<u>=</u>
Power	1E Bus A or B	4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4
Valve	Closure Time (s)	<u> </u>	<u> </u>
	Actuation Signal	55 \$ 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	<u></u>
	Power Failure	<u>₹</u> \$	<u>₹</u> \$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$
LO.	Post- Accident	000000000000000000000000000000000000000	000000000000000000000000000000000000000
Valve Position	Shutdown		
>	Normal SI	000000000000000000000000000000000000000	000000000000000000000000000000000000000
ı İ	21		
opol	Secondary	Agenole man. Vone Vone Vone Vone Vone Vone Vone Vone	Remote man. None None None None None None None None
Actuation Mode	Š	222222222222222222222222222222222222222	222222222222222222222222222222222222222
4	Primary	Auto Manual Marual Marual Manual	Auto Manual Auto Auto Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual
	Essential or Nonessential		
1	Esse	mzzzzzzzzzzzzzzzzz	mzzzzzzzzzzzzzzz
Valve	Operator	Elec. motor Manual Manu	Elec. motor Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual
	Type	B-fly Gate Relief Globe	B-fly Globe
Ę.	. 7	_	
Length		5.0	.0-
	t Type Tests	∢	∢
Location Relative to Con-	tainment Inside/ Outside	ğ	ğ
	Valve	HV-1806 002 1002 1003 1004 1006 1006 1006 1006 1006 1006 1006	HV-1806 0015 0015 PSV-1816 X-253 X-193 X-192 X-192 X-199 X-199 X-191 Z-190 Z-1
Valve	Arrange- ment Fig. 6.2.4-1		
- Va	Mr. 6.2	44	55 - 25
	Drawing Number	2X4DB135-1 2X4DB135-1	1X4DB135-1 2X4DB135-1
	ESF or Support Systems	Yes	, Yes
:	Size (in.)	8 0.75 0.05 0.05 0.05 0.05 1.11 1.11	8 0.75 0.5 0.5 0.5 0.5 0.75 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	Fluid	Treated well water	Treated well water
	System Name	NSCW supply to	NSOW supply to
	GDC or RG	00 00 Line 1	00 00 00 00 00 00 00 00 00 00 00 00 00
	Penetra- tion (Number or		
1	₹ § ₹	83	8

TABLE 6.2.4-1 (SHEET 15 OF 17)

	Normal Direction of Flow	o o o	one	Ont
Power	1E Bus A or B	m 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	u 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	< 222222222222222222222222222222222222
Valve	Closure Time (s)	<u> </u>	22222222222222	<u>\$</u> \$\$\$\$\$\$\$\$\$\$\$\$\$
	Power Actuation Failure Signal	552222222222222 53222222222222222222222	\$\frac{1}{2}\frac{1}\frac{1}{2}\f	55 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5
	Post- F Accident F	#52555555555		# Z Z Z Z Z Z Z Z Z Z Z Z Z Z Z Z Z Z Z
Valve Position		000000000000000	0000000000000000	000000000000
Valve	al Shutdown	000000000000000	00000000000000000	00000000000
	Normal	000000000000000	00000000000000000	00000000000
Actuation Mode	Secondary	Norme man Norme man Norme man Norme man Norme more more more norme norme norme norme norme norme norme norme norme norme norme more more more more more more more	N N N N N N N N N N N N N N N N N N N	Remote man. None None None None None None None None
Ad	Primary	Auto Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual	Auto Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual	Auto Manual Manual Manual Manual Manual Manual Manual Manual
	=	42222222222	4222422222222	422222222
ı	Essential or Nonessential	шZZZZZZZZZZZZZ	mzzzzzzzzzzzzzz	m z z z z z z z z z z
Valve	Operator	Elec. motor Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual Manual	Elec. motor Manual	Elec. motor Manual Manual Manual Manual Manual Manual Manual Manual
	Type	B-ffy G G G G G G G G G G G G G G G G G G	P-ffy G G G G G G G G G G G G G G G G G G G	B-fly Globe Globe Globe Globe Globe Globe Globe Globe
-ength	of Pipe (ff-in.)	22-0"	1-6"	2.0"
_	Type F			
Location Relative to Con-	-	∢	∢	∢
Locatior Relative to Con-	tainmen Inside/ Outside	900 555555555555	 Agenenenenenenen	Q = = = = = = = = =
	g. Valve Number	HV-1831 C-046 034 034 X-489 X-383 X-383 X-383 X-383 X-384 3-67 3-73	HV-1823 X-501 (o) X-501 (o) X-501 (o) X-501 (o) X-501 (o) X-501 (o) 401 401 417 (o) 417 (o) 419 419 419 411 (o) 419 411 (o) 419 411 (o) 411 (o) 411 (o) 412 (o) 413 (o) 414 (o) 415 (o) 416 (o) 417 (o) 418 (o) 4	HV-1830 017 X-177 X-178 263 265 271 277 275 275 275 275 275 275 275 275 275
Valve	Arrange- ment Fig. 6.2.4-1	9	9	9
	Drawing	1X4DB135-2	1X4DB135-2 2X4DB135-2	1X4DB135-1 2X4DB135-1
	ESF or Support Systems	Yes	Yes	Xes
	Line Size (in.)		8 <u>0</u>	
	Fluid	Treated well water	Treated well water	Treated well water
	System Name	NSCW return from containment coolers	NSCW return from confairment coolers	NSOW return from containment coolers
	a- GDC	57	57	57
	Penetra- tion Number	99	8	16

TABLE 6.2.4-1 (SHEET 16 OF 17)

Norman	Direction of Flow	O et	Out	<u>=</u>	<u>=</u>
Power Source	Bus A	< < < < < < < < < < < < < < < < < < <	Y M Y Y	4 4 4 ⁰ 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	4 4 4 ⁰ 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4
Valve	Time (s)	<u> </u>	NAN NAN	ian. (f)	ran (h).
	Actuation Signal	<u>~</u> \$\$\$\$\$\$\$\$\$\$\$\$	2544 2554	NA NA NA NA Remote man. NA NA	NA NA NA NA NA NA NA NA NA
	Power	<u>₹</u> \$\$\$\$\$\$\$\$\$\$\$\$\$	<u> </u>	\$ \$\$0\$\$0\$\$\$\$	444044044
	Post- Accident		0022	0	900 .0000
Valve Position	Shutdown	00000000000		,,900,00000	,,400,0000
Valv	Normal Shut	00000000000	CCCC	, ,900,00000	, ,900,0000
	Nor	00000000000	2220	, ,900,00000	, ,900,0000
Actuation Mode	Secondary	Remote man. None None None None None None None None	Remote man. Remote man. None	None None Remole man. None None None None None None None None	None None None Remote man. None None None None None None
Actu	Primary	Auto Manual Manual Manual Manual Manual Manual Manual Manual	Auto Auto Manual Manual	Auto Auto Auto Auto Auto Manual Auto Manual Manual Manual	Auto Auto Auto Auto Manual Auto Remote man. Manual Manual
ı	Essential or Nonessential	шZZZZZZZZZZ	zzzz	шZшZZZZZZZ	шZшZZZZZZ
Valve	Operator	Elec. motor Manual Manual Manual Manual Manual Manual Manual Manual	Elec. motor Elec. motor Manual Manual	Seff Seff Seff Air Manual Manual Manual Manual	Seff Seff Seff Air Manual Seff Air Manual Manual
	Type	B-fly Globe Globe Globe Globe Globe Globe Globe Globe Globe Globe	B-fly B-fly Gate Gate	Check Check Stop Gate Globe Globe Globe Globe Globe Globe	Check Check Stop Gate Globe Globe Globe Globe
Length	(f-in.)		6'-6"	5.28 8.13 8.13 1.18 18.19 31.44	6'-4" 6'-6" 9'-0" 2'-9" 8'-0" 30'-7"
	Type Tests	∢	O	<	<
Location Relative to Con-	Inside/ Outside	ğ	ont the	n 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	1 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
	Valve	HV-1822 016 X-179 289 287 295 301 299 307 342 342 344 X-180	HV-2624A HV-2624B 012 001	128 120 115 115 115 136 136 136 136 136 136 136 136 136 136	126 118 114 114 HV-15197 X-195 134 HV-5195 X-188 X-197 X-237
Valve	ment Fig. 6.2.4-1	91	22	64	49
	Drawing Number	1X4DB135-1 2X4DB135-1	1X4DB213-1 2X4DB213-1	1X4DB168-3 2X4DB168-3	1X4DB168-3 2X4DB168-3
0	Support	Yes	° Z	Yes	Yes
<u></u>	Size (in.)	&	4 4 0.75	9 9 4 9 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6
	Fluid	Treated well water	Containment atmosphere	Secondary	Secondary
	System Name	NSCW return from containment coolers	Post-accident air exhaust	Auxilary feedwater	Auxiiary feedwater
	or RG	57	26	57(1)	924(1)
Gonatra	tion	8	100	101	102

TABLE 6.2.4-1 (SHEET 17 OF 17)

	_ 5 -				
	Normal Direction of Flow	٩	<u>=</u>		
Power		~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~	~ < < [©] < < < < < < < < < < < < < < < < < < <	₹ ₹ Z Z	NA d by the
Valve	Closure Time (s)	\$\$\$\$\$\$\$\$\$\$ £.	\$\$\$\$\$\$\$\$ £:	≨ ≨	C C C NA NA NA NA NA NA THE STOKE THE STOKE THE STOKE THE OF THE SPECIFICATIONS.
	Actuation Signal	NA NA NA FI NA NA NA NA NA NA	NA NA NA NA NA NA NA	§ §	NA penetration is
	Power	₹ ₹₽₹₹₽₹₹	₹ ₹₽₹₹₽₹	§ §	NA 0 s. This
5	Post- Accident	, ,900,000	, ,900,00	0 0	C values is 2
Valve Position	Shutdown	0	. ,900,00	υυ	C C The stroke time of these
	Normal	200 . 000	900.00	0 0	C C e stroke time chnical Spe
ė	Secondary	te man.	le man.	al al	÷
Actuation Mode	Secor			Manual	Manual
Act	Primary	Auto Auto Auto Auto Auto Remote man Manual Manual	Auto Auto Auto Manual Auto Remote man. Manual	Manual	Manual
	Essential or Nonessential	wzwzzzzzz	wzwzzzzz	z z	z
Valve	Operator	Seff Seff Seff Air Manual Seff Manual Manual	Self Self Self Air Manual Self Air	None None	None
>	Type	Check Check Stop Gate Globe Globe Globe Globe	Check Check Stop Gate Globe Check Globe	None None	None w-low.
Length	of Pipe (ft-in.)	130" 10'-9" 15'-10" 5'-2" 7'-1"	3-9" 9-1" 17-2" 5-2" 7-1"	₹ ₹	NA B NA Non Opens coincident on SI and RWST low-low.
	Type Tests	<	<	ш ш	B ident on S
Location Relative to Con-	tainment Inside/ Outside	O O O O O O O O O O O O O O O O O O O	- 0000000 u	§ §	
	Valve	127 119 116 HV-15199 X-180 ^(P) 135 HV-5197 X-215 X-215	125 117 113 113 114V-15196 X-178 133 14V-5194 X-233	A A	e K Z
Valve	Arrange- ment Fig. 6.2.4-1	64	49	4 4 4 3 4 3	84
	Drawing Number	2X4DB168-3	1X4DB168-3 2X4DB168-3	\$ \$	Ž
	ESF or Support Systems	Yes	Yes	0	o Z
	Size S	6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	10.10	z z § §	Z Y
	⊐ Ø ⊖				
	Fluid	Secondary	Secondary coolant	Containment atmosphere Containment atmosphere	Containment Containment S:
	System Name	Auxiliary feedwater	57(i) Auxiliary feedwater	Equipment hatch Personnel locks	56 Emergency doors a. The following is a list of abbreviations:
	GDC or RG	o 22(i) v	, (j) 224(i)	56 56 F	56 E
	Penetra- tion Number	103	104		a. The fol

Valve dosure time is defined in SRP 6.2.4 paragraph II.6.N.

The ACCW penetrations are not part of an engineered safety features system nor are they required for safe shutown, however, the penetrations are desirided as essential due to the importance of maintaining counting water to the reactor contant puring. These valves remain open post-accident and are only closed for certain accident conditions. ė,

Power to the solenoid shall be removed during all modes of operation, except during wet is upon operation and operation and provide testing for the valves, in younging the side links of the states it reminal bods, to lead on the open position. Ė

steam free includion feed-water leolation containment ventilation isolation or containment isolation phase A containment isolation phase B safety bjockon signal selectroyyaulic feed-oring and includion feed-oring and includion audian's feed-water automatic start signal

SLI CVI CVI CVI SSI SSI SSI SAI AFS

open closed closed closed falled closed actuated by the fluid pressure not applicable

General Design Criteria NRC Regulatory Guides diaphragm

butterfly

GDC RG Dia Dia C C C C C C F C Seff NA Auto

These penetrations are associated with the secondary side of the steam generators. The present design belation proteiors for secondary system interes meet the first of GOD 57 (i.e., or solidor valve capable of remote manual operation is provided), but GOD 57 was not the design beast. The steam generator and associated secondary system point from the design beast. The steam generator and associated secondary system point from the design permiss that the same as the containment liner plate. The valves associated with these penetrations do not receive a containment analysis. The valves associated with affecting containment isolation signal and are not credited with affecting containment isolation is gone and care not capable.

These valves are required to function for long term codedown and recirculation. These valves and/or finges are opened for the refueling operation. These valves identified in risk table are shown with an asterisk (') in figure 6.2.4-1.

o o o

These penetrations are associated with the steam generator eddy current and sludge landing generators. During modes 6 and 6, the linfel flanges may be removed and relabaced with a facult of the allows cables and hoses to pass through. The cables are sealed, and manually operated valves are provided on the hoses both inside and outside containment to provide the capability for containment closure during fuel movement as required by predifications. ×.

Valve disk is provided with bonnet vent on the containment side of disk.

m. See table 6.3.2-3 for other stroke time information.

For LOCA conditions with a loss of one train of dc power, the secondary actuation mode for views HV-2784AB to a recomplished by providing an alternate train power connection in the QPOP partle per plant procedures.

Unit 1 only.

Unit 2 only.

Unit 2 is a gate valve.

r. 7 s for system pressure > 325 psia; 40 s for system pressure ≤ 325 psia. Note r applicable to Unit 2 only.

Train A only.

TABLE 6.2.4-2 (SHEET 1 OF 15)

CONTAINMENT ISOLATION VALVES(a)

	Valve <u>Number</u>	<u>Function</u>	Valve Closure <u>Time(s)</u>
1.	Containment Isolat	ion Phase "A"	
	1HV-3502	Hot leg sample line	≤ 15
	1HV-3548	Hot leg sample line	≤ 20
	2HV-3502	Hot leg sample line and gross failed fuel detector	≤ 15
	2HV-3548	Hot leg sample line and gross failed fuel detector	≤ 20
	HV-8823	Safety injection pump discharge to cold leg	≤ 15
	HV-8824	Safety injection pump discharge to hot leg	≤ 15
	HV-8843	Boron injection line to cold leg	≤ 15
	HV-8881	Safety injection pump discharge to hot leg	≤ 15
	HV-27901	Fire protection water	≤ 20
	HV-8871	Accumulator test and drain line	≤ 15
	HV-8964	Accumulator test and drain line	≤ 15
	HV-8888	Accumulator test and fill line	≤ 15
	HV-8880	Nitrogen supply to accumulator	≤ 15
	HV-8160	Normal letdown line	≤ 15

TABLE 6.2.4-2 (SHEET 2 OF 15)

	Valve <u>Number</u>	<u>Function</u>	Valve Closure <u>Time(s)</u>
1.	Containment Isolati	on Phase "A" (continued)	
	HV-8152	Normal letdown line	≤ 15
	HV-8100	Excess letdown and seal water leakoff	≤ 15
	HV-8112	Excess letdown and seal water leakoff	≤ 15
	HV-8825	RHR pump discharge to hot leg	≤ 15
	HV-8890A	RHR pump discharge to cold leg	≤ 15
	HV-8890B	RHR pump discharge to cold leg	≤ 15
	HV-8033	Pressurizer relief tank sample to waste gas compressor suction	≤ 15
	HV-8047	Pressurizer relief tank sample to waste gas compressor suction	≤ 15
	HV-8028	Pressurizer relief tank makeup water supply	≤ 15
	HV-3514	Pressurizer steam sample line	≤ 15
	HV-3513	Pressurizer steam sample line	≤ 15
	HV-3507	Pressurizer liquid sample line	≤ 15
	HV-3508	Pressurizer liquid sample line	≤ 15
	HV-10950	Accumulator sample line	≤ 15
	HV-10952	Accumulator sample line	≤ 15
	HV-10951	Accumulator sample line	≤ 15
	HV-10953	Accumulator sample line	≤ 15
	HV-7699	Reactor coolant drain tank pump discharge	≤ 15

TABLE 6.2.4-2 (SHEET 3 OF 15)

	Valve <u>Number</u>	<u>Function</u>	Valve Closure <u>Time(s)</u>
1.	Containment Isolat	ion Phase "A" (continued)	
	HV-7136	Reactor coolant drain tank pump discharge	≤ 15
	HV-0780	Normal containment sump pumps discharge	≤ 15
	HV-0781	Normal containment sump pumps discharge	≤ 15
	HV-7126	Reactor coolant drain tank vent and H ₂ supply	≤ 15
	HV-7150	Reactor coolant drain tank vent and H ₂ supply	≤ 15
	HV-9385	Service air and post-LOCA purge air supply	≤ 20
	HV-9378	Instrument air	≤ 15
	HV-8211	Post-accident sampling	≤ 15
	HV-8212	Post-accident sampling	≤ 15
2.	Containment Ventil	ation Isolation	
	HV-12975	Containment air radioacitvity monitor inlet	≤ 15
	HV-12976	Containment air radioactivity monitor inlet	≤ 15
	HV-12977	Containment air radioactivity monitor outlet	≤ 15
	HV-12978	Containment air radioactivity monitor outlet	≤ 15
	HV-2626A	Containment pre-access purge supply and equalizing	≤ 10
	HV-2627A	Containment pre-access purge supply and equalizing	≤ 10

TABLE 6.2.4-2 (SHEET 4 OF 15)

	Valve		Valve Closure
	Number	<u>Function</u>	Time(s)
2.	Containment Ventil	ation Isolation (continued)	
	HV-2626B	Containment mini-purge supply and equalizing	≤ 5
	HV-2627B	Containment mini-purge supply and equalizing	≤ 5
	HV-2628A	Containment pre-access purge exhaust and equalizing	≤ 10
	HV-2629A	Containment pre-access purge exhaust and equalizing	≤ 10
	HV-2628B	Containment mini-purge exhaust and equalizing	≤ 5
	HV-2629B	Containment mini-purge exhaust and equalizing	≤ 5
	HV-2624A	Post-accident air exhaust	N/A
	HV-2624B	Post-accident air exhaust	N/A
3.	Safety Injection		
	HV-8811B ^(d)	RHR emergency sump suction	N/A
	HV-8811A ^(d)	RHR emergency sump suction	N/A
	HV-2134 ^(e)	NSCW supply to reactor cavity coolers	≤ 40
	HV-2138 ^(e)	NSCW return from reactor cavity coolers	≤ 40
	HV-2135 ^(e)	NSCW supply to reactor cavity coolers	≤ 40
	HV-2139 ^(e)	NSCW return from reactor cavity coolers	≤ 40
	HV-8105	Normal charging line	≤ 17 ^(j)
	HV-1809 ^(e)	NSCW supply to containment coolers	N/A

TABLE 6.2.4-2 (SHEET 5 OF 15)

	Valve <u>Number</u>	<u>Function</u>	Valve Closure <u>Time(s)</u>
3.	Safety Injection (co	ontinued)	
	HV-1807 ^(e)	NSCW supply to containment coolers	N/A
	HV-1806 ^(e)	NSCW supply to containment coolers	N/A
	HV-1808 ^(e)	NSCW supply to containment coolers	N/A
	HV-1831 ^(e)	NSCW return from containment coolers	N/A
	HV-1823 ^(e)	NSCW return from containment coolers	N/A
	HV-1830 ^(e)	NSCW return from containment coolers	N/A
	HV-1822 ^(e)	NSCW return from containment coolers	N/A
	HV-8801A ^(d)	Boron injection line to cold leg	N/A
	HV-8801B ^(d)	Boron injection line to cold leg	N/A
4.	Check Valves		
	1418-U4-038	Demineralized water supply	N/A
	2401-U4-184	Breathing air supply	N/A
	1217-U4-113	ACCW return	N/A
	1204-U4-143	Safety injection to cold leg	N/A
	1204-U4-144	Safety injection to cold leg	N/A
	1204-U4-145	Safety injection to cold leg	N/A
	1204-U4-146	Safety injection to cold leg	N/A
	1204-U4-122	Safety injection to hot leg	N/A
	1204-U4-123	Safety injection to hot leg	N/A
	1204-U6-013	Boron injection to cold leg	N/A

TABLE 6.2.4-2 (SHEET 6 OF 15)

	Valve <u>Number</u>	<u>Function</u>	Valve Closure <u>Time(s)</u>
4.	Check Valves (cont	inued)	
	1204-U4-120	Safety injection to hot leg	N/A
	1204-U4-121	Safety injection to hot leg	N/A
	1206-U6-016	Containment spray supply	N/A
	1206-U6-015	Containment spray supply	N/A
	2301-U4-036	Fire protection water	N/A
	2402-U4-017	Nitrogen supply to accumulator	N/A
	1208-U4-021	Excess letdown and seal water leakoff	N/A
	1208-U6-032	Normal charging line	N/A
	1208-U4-355	Reactor coolant pump seal water supply	N/A
	1208-U4-354	Reactor coolant pump seal water supply	N/A
	1208-U4-353	Reactor coolant pump seal water supply	N/A
	1208-U4-004	Reactor coolant pump seal water supply	N/A
	1204-U6-128	RHR pump discharge to hot leg	N/A
	1204-U6-129	RHR pump discharge to hot leg	N/A
	1204-U6-147	RHR loop into cold leg	N/A
	1204-U6-148	RHR loop into cold leg	N/A
	1204-U6-149	RHR loop into cold leg	N/A
	1204-U6-150	RHR loop into cold leg	N/A
	1201-U6-112	Pressurizer relief tank makeup water supply	N/A
	1513-U4-001	Containment H ₂ monitor discharge	N/A

TABLE 6.2.4-2 (SHEET 7 OF 15)

	Valva		Valve
	Valve <u>Number</u>	<u>Function</u>	Closure Time(s)
4.	Check Valves (con	tinued)	
	1513-U4-002	Containment H ₂ monitor discharge	N/A
	2401-U4-034	Service air and post-LOCA purge air supply	N/A
	2420-U4-049	Instrument air	N/A
	1302-U4-126 ^(k)	Auxiliary feedwater	N/A
	1302-U4-118 ^(k)	Auxiliary feedwater	N/A
	1302-U4-114 ^(k)	Auxiliary feedwater	N/A
	1302-U4-134 ^(k)	Auxiliary feedwater	N/A
	1302-U4-128 ^(k)	Auxiliary feedwater	N/A
	1302-U4-120 ^(k)	Auxiliary feedwater	N/A
	1302-U4-115 ^(k)	Auxiliary feedwater	N/A
	1302-U4-136 ^(k)	Auxiliary feedwater	N/A
	1302-U4-127 ^(k)	Auxiliary feedwater	N/A
	1302-U4-119 ^(k)	Auxiliary feedwater	N/A
	1302-U4-116 ^(k)	Auxiliary feedwater	N/A
	1302-U4-135 ^(k)	Auxiliary feedwater	N/A
	1302-U4-125 ^(k)	Auxiliary feedwater	N/A
	1302-U4-117 ^(k)	Auxiliary feedwater	N/A
	1302-U4-113 ^(k)	Auxiliary feedwater	N/A
	1302-U4-133 ^(k)	Auxiliary feedwater	N/A

TABLE 6.2.4-2 (SHEET 8 OF 15)

	Valve <u>Number</u>	<u>Function</u>	Valve Closure <u>Time(s)</u>
5.	Remote Manual		
	HV-5280 ^(b)	Chemical Addition	N/A
	HV-5281 ^(b)	Chemical Addition	N/A
	HV-1978	ACCW Supply	N/A
	HV-1979	ACCW Supply	N/A
	HV-1974	ACCW Return	N/A
	HV-1975	ACCW Return	N/A
	HV-8835	Safety injection to cold leg	N/A
	HV-8802B	Safety injection to hot leg	N/A
	HV-8802A	Safety injection to hot leg	N/A
	HV-9002B ^(f)	Containment spray emergency sump suction	N/A
	HV-9002A ^(f)	Containment spray emergency sump suction	N/A
	HV-8103D	Reactor coolant pump seal water supply	N/A
	HV-8103B	Reactor coolant pump seal water supply	N/A
	HV-8103C	Reactor coolant pump seal water supply	N/A
	HV-8103A	Reactor coolant pump seal water supply	N/A
	HV-8840	RHR pump discharge to hot leg	N/A
	HV-8809A	RHR loop into cold leg	N/A
	HV-8809B	RHR loop into cold leg	N/A
	HV-8701A	RHR suction from hot leg	N/A
	HV-8702A	RHR suction from hot leg	N/A

TABLE 6.2.4-2 (SHEET 9 OF 15)

	Valve <u>Number</u>	<u>Function</u>	Valve Closure <u>Time(s)</u>
5.	Remote Manual (co	ontinued)	
	HV-5278 ^(b)	Chemical addition	N/A
	HV-5279 ^(b)	Chemical addition	N/A
	HV-2791A ⁽ⁱ⁾	Containment H ₂ monitor suction	N/A
	HV-2790A ⁽ⁱ⁾	Containment H ₂ monitor suction	N/A
	HV-2790B ⁽ⁱ⁾	Containment H ₂ monitor suction	N/A
	HV-2793A ⁽ⁱ⁾	Containment H ₂ monitor discharge	N/A
	HV-2792B ⁽ⁱ⁾	Containment H ₂ monitor suction	N/A
	HV-2791B ⁽ⁱ⁾	Containment H ₂ monitor suction	N/A
	HV-2792A ⁽ⁱ⁾	Containment H ₂ monitor suction	N/A
	HV-2793B ⁽ⁱ⁾	Containment H ₂ monitor discharge	N/A
	HV-5194 ^{(b)(k)}	Auxiliary feedwater	N/A
	HV-5197 ^{(b)(k)}	Auxiliary feedwater	N/A
	HV-5195 ^{(b)(k)}	Auxiliary feedwater	N/A
	HV-5196 ^{(b)(k)}	Auxiliary feedwater	N/A
	HV-9556A ^(k)	Steam generator secondary side sample	N/A
	HV-9556B ^(k)	Steam generator secondary side sample	N/A
	HV-9555A ^(k)	Steam generator secondary side sample	N/A
	HV-9555B ^(k)	Steam generator secondary side sample	N/A
	HV-9554A ^(k)	Steam generator secondary side sample	N/A
	HV-9554B ^(k)	Steam generator secondary side sample	N/A

TABLE 6.2.4-2 (SHEET 10 OF 15)

	Valve <u>Number</u>	<u>Function</u>	Valve Closure <u>Time(s)</u>
5.	Remote Manual (co	ontinued)	
	HV-9553A ^(k)	Steam generator secondary side sample	N/A
	HV-9553B ^(k)	Steam generator secondary side sample	N/A
	HV-3009 ^(h)	Main steam to auxiliary feedwater pump driver	N/A
	HV-3019 ^(h)	Main steam to auxiliary feedwater pump driver	N/A
6.	<u>Manual</u>		
	1213-U6-050 ^(c)	Purification water supply to refueling cavity	N/A
	1213-U6-051 ^(c)	Purification water supply to refueling cavity	N/A
	1418-U4-005 ^(c)	Demineralized water supply	N/A
	2401-U4-211 ^(c)	Breathing air supply	N/A
	1411-U4-676 ^(c)	Chemical addition	N/A
	1411-U4-677 ^(c)	Chemical addition	N/A
	1411-U4-678 ^(c)	Chemical addition	N/A
	1411-U4-679 ^(c)	Chemical addition	N/A
	1204-U4-159 ^(c)	Accumulator sample line	N/A
	1204-U4-161 ^(c)	Accumulator sample line	N/A
	1204-U4-160 ^(c)	Accumulator sample line	N/A
	1204-U4-162 ^(c)	Accumulator sample line	N/A
	1202-U4-001	NSCW supply to containment fire protection	N/A
	1202-U4-002	NSCW supply to containment fire protection	N/A
	1508-U4-012 ^(c)	Post-accident air exhaust	N/A

TABLE 6.2.4-2 (SHEET 11 OF 15)

	Valve <u>Number</u>	<u>Function</u>	Valve Closure <u>Time(s)</u>
7.	Containment Spray	<u></u>	
	HV-9001A	Containment spray supply	N/A
	HV-9001B	Containment spray supply	N/A
8.	Pressure Relief Va	lves	
	PSV-17589	Plant demineralized water to containment	N/A
	PSV-11673	NSCW supply to reactor cavity coolers	N/A
	PSV-2136	NSCW from reactor cavity coolers	N/A
	PSV-2137	NSCW supply to reactor cavity coolers	N/A
	PSV-11772	NSCW from reactor cavity coolers	N/A
	PSV-8708A	RHR suction from hot leg	N/A
	PSV-8708B	RHR suction from hot leg	N/A
	PSV-1817	NSCW supply to containment coolers	N/A
	PSV-11774	NSCW supply to containment coolers	N/A
	PSV-1815	NSCW supply to containment coolers	N/A
	PSV-1814	NSCW supply to containment coolers	N/A
	PSV-11672	NSCW supply to containment coolers	N/A
	PSV-1816	NSCW supply to containment coolers	N/A
	PSV-11671	NSCW supply to containment coolers	N/A
	PSV-11773	NSCW supply to containment coolers	N/A
	PSV-3001	Main steam	N/A
	PSV-1978	ACCW Supply	N/A

TABLE 6.2.4-2 (SHEET 12 OF 15)

	Valve <u>Number</u>	<u>Function</u>	Valve Closure <u>Time(s)</u>			
8.	Pressure Relief Val	Pressure Relief Valves (continued)				
	PSV-8871	Accumulator Test and Drain	N/A			
	PSV-7699	RCDT Pump Discharge	N/A			
	PSV-0780	Normal Containment Sump Pump	N/A			
	PSV-3002	Main steam	N/A			
	PSV-3003	Main steam	N/A			
	PSV-3004	Main steam	N/A			
	PSV-3005	Main steam	N/A			
	PSV-3011	Main steam	N/A			
	PSV-3012	Main steam	N/A			
	PSV-3013	Main steam	N/A			
	PSV-3014	Main steam	N/A			
	PSV-3015	Main steam	N/A			
	PSV-3021	Main steam	N/A			
	PSV-3022	Main steam	N/A			
	PSV-3023	Main steam	N/A			
	PSV-3024	Main steam	N/A			
	PSV-3025	Main steam	N/A			
	PSV-3031	Main steam	N/A			
	PSV-3032	Main steam	N/A			
	PSV-3033	Main steam	N/A			

TABLE 6.2.4-2 (SHEET 13 OF 15)

	Valve <u>Number</u>	<u>Function</u>	Valve Closure <u>Time(s)</u>	
8.	Pressure Relief Val	<u>lves</u> (continued)		
	PSV-3034	Main steam	N/A	
	PSV-3035	Main steam	N/A	
9.	Other Automatic Va	<u>alves</u>		
	HV-3006A ^(g)	Main steam	≤5 (Unit 1), Note m	
	HV-3006B ^(g)	Main steam	≤5	
	HV-3016A ^(g)	Main steam	≤5 (Unit 1), Note m	
	HV-3016B ^(g)	Main steam	≤5	
	HV-13005A ^(g)	Main steam	≤5	
	HV-13005B ^(g)	Main steam	≤5	
	HV-13007A ^(g)	Main steam	≤5	
	HV-13007B ^(g)	Main steam	≤5	
	HV-3026A ^(g)	Main steam	≤5 (Unit 1), Note m	
	HV-3026B ^(g)	Main steam	≤5	
	HV-13008A ^(g)	Main steam	≤5	
	HV-13008B ^(g)	Main steam	≤5	
	HV-3036A ^(g)	Main steam	≤5 (Unit 1), Note m	
	HV-3036B ^(g)	Main steam	≤5	
	HV-13006A ^(g)	Main steam	≤5	
	HV-13006B ^(g)	Main steam	≤5	

TABLE 6.2.4-2 (SHEET 14 OF 15)

	Valve <u>Number</u>	<u>Function</u>	Valve Closure <u>Time(s)</u>				
9.	Other Automatic \	Other Automatic Valves (continued)					
	HV-7603A ^(h)	Steam generator blowdown	≤15				
	HV-7603B ^(h)	Steam generator blowdown	≤15				
	HV-7603C ^(h)	Steam generator blowdown	≤15				
	HV-7603D ^(h)	Steam generator blowdown	≤15				
	HV-9451 ^(h)	Steam generator secondary side sample	≤15				
	HV-9452 ^(h)	Steam generator secondary side sample	≤15				
	HV-9453 ^(h)	Steam generator secondary side sample	≤15				
	HV-9454 ^(h)	Steam generator secondary side sample	≤15				
	HV-5229 ^(g)	Feedwater	≤5				
	HV-5228 ^(g)	Feedwater	≤5				
	HV-5230 ^(g)	Feedwater	≤5				
	HV-5227 ^(g)	Feedwater	≤5				
	HV-15198 ^(g)	Feedwater	≤5				
	HV-15197 ^(g)	Feedwater	≤5				
	HV-15196 ^(g)	Feedwater	≤5				
	HV-15199 ^(g)	Feedwater	≤5				
	PV-3000	Main steam	N/A				
	PV-3010	Main steam	N/A				
	PV-3020	Main steam	N/A				
	PV-3030	Main steam	N/A				

TABLE 6.2.4-2 (SHEET 15 OF 15)

- a. See FSAR subsection 6.2.4 for discussion of the containment isolation system.
- b. The containment isolation valves will be maintained closed by administratively controlling the air supply valve links in an open position.
- c. Locked closed.
- d. These valves are included for table completeness. The requirements of Technical Specification 3.6.3 do not apply; instead the requirements of Technical Specification 3.5.2 apply. Valve stroke times where specified will be tested pursuant to the IST program.
- e. These valves are included for table completeness. The requirements of Technical Specification 3.6.3 do not apply; instead the requirements of Technical Specification 3.7.8 apply. Valve stroke times where specified will be tested pursuant to the IST program.
- f. These valves are included for table completeness. The requirements of Technical Specification 3.6.3 do not apply; instead the requirements of Technical Specification 3.6.6 apply. Valve stroke times where specified will be tested pursuant to the IST program.
- g. These valves are included for table completeness. The requirements of Technical Specification 3.6.3 do not apply; instead the requirements of Technical Specification 3.7.2 apply to the main steam isolation and bypass valves and Technical Specifications 3.3.2 and 3.7.3 apply to the main feedwater isolation valves.
- h. These valves are included for table completeness. The requirements of Technical Specification 3.6.3 do not apply; instead the requirements of Technical Specification 3.7.5 apply. Valve stroke times where specified will be tested pursuant to the IST program.
- i. These valves may be opened on an intermittent basis under administrative control.
- j. See table 6.3.2-3 for other stroke time information.
- k. These valves are included for table completeness. The requirements of Technical Specification 3.6.3 do not apply; instead verification of containment isolation is accomplished in the type A leakage test which credits the steam generator tubes and associated piping as the primary barrier to the outside environment. These valves are associated with the secondary side of the steam generators and are not subject to GDC 57. These valves do not receive a containment isolation signal and are not credited with effecting containment isolation in the safety analyses. Reference UFSAR paragraph 6.2.4.2.1 and table 6.2.4-1, penetrations 102, 103, and 104 and reference note "I."

NOTE: The valve table as shown reflects the inservice test program as modified by the second 10-year interval update. The updated IST program will be fully implemented by May 31, 1998.

m. 7 seconds for system pressure >325 psia; 40 seconds for system pressure ≤325 psia. Note m applicable to Unit 2 only.

TABLE 6.2.5-1

DESIGN DATA FOR HYDROGEN RECOMBINERS

Quantity 2 per unit Power (each) (maximum/minimum) (kW) 75/50 Capacity (each) (minimum) (sf³/min) 100

Heaters (per recombiner)

Number 4 banks Maximum heat flux (Btu/h-ft²) 2850 Maximum sheath temperature (°F) 1550

Gas temperatures

Inlet (°F) Outlet of heater section (°F)

Exhaust (°F)

Materials

Outer structure Type 300 series SS Incoloy 800 Inner structure

Heater element sheath

Base skid Weight (lb)

Codes and standards

above ambient

Incoloy 800

Type 300 series SS

4500 lb

80-155

1150 to 1400 Approximately 50

American Society of **Mechanical Engineers** Section IX, Underwriters Laboratory, **National Electric** Manufacturers Association, **National Fire** Protection Association, Institute of Electrical and

Engineers 279, 308, 323, 344, and 383, Safety Class 0 (See table 3.2.2-1.)

Electronic

TABLE 6.2.5-2 (SHEET 1 OF 2)

CONTAINMENT BUILDING - FAILURE MODES AND EFFECTS ANALYSIS, POST-LOCA CAVITY PURGE SYSTEM

Failure Effect on System Safety Function Capability	None; loss of train A; train B available	None; loss of train A; train B available		None; loss of train A; train B available	None; loss of train A; train B available	None; loss of train B; train A available	Flow alarm low	None; loss of train B; train A available		None; loss of train B; train A available
Method of Failure <u>Detection</u>	Motor control center (MCC) alarm Fan motor lights Flow alarm low	Fan motor lights	Flow alarm low	Flow alarm low	Flow alarm low	MCC alarm	Fan motor lights	Fan motor lights	Flow alarm low	Flow alarm low
Failure <u>Mode(s)</u>	Inadvertent open	Fail to close		Fail to start and operate	Fail to open	Inadvertent open		Fail to close		Fail to start and operate
Plant Operating <u>Mode^(a)</u>	∢	∢		4	⋖	∢		∢		∢
Safety <u>Function</u>	Provide continuity and protection to fan motor, item 3	Provide continuity to fan motor, item 3		Provide motive power by supplying air to the reactor cavity	Allow flow of air to the reactor cavity and prevent backflow	Provide continuity and protection to	1000, 1000	Provide continuity	נט ומון וווטנטו, ונפוון ל	Provide motive power by supplying air to the reactor cavity
Description of Component	No. 29 breaker on 1ABE, 480-V, 1E, MCC, train A normally closed (NC)	No. 29 motor starter, for item 3, train A normally open (NO)		1-1516-B7-001-M01, Containment building (CTB) post-LOCA cavity purge unit motor and fan train A, normally deenergized (ND)	003 backflow damper, train A, NC	No. 29 breaker on 1BBE, 480-V, 1E, MCC, train B, NC		No. 29 motor starter, item 7 train	O. C.	1-1516-B7-002-M01 CTB post- LOCA cavity purge unit motor and fan, train B, NO
Item No.	-	2		ю	4	ιO		9		-

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TABLE 6.2.5-2 (SHEET 2 OF 2)

Failure Effect on System Safety Function Capability	None; loss of train B; train A available	None; loss of train A; train B available		None; loss of train B; train A available
Method of Failure <u>Detection</u>	Flow alarm low	Flow alarm, low	Motor indicating light	Flow alarm low Motor indicating light
Failure <u>Mode(s)</u>	Fail to open	Mechanical failure		Mechanical failure
Plant Operating Mode ^(a)	⋖	⋖		∢
Safety <u>Function</u>	Allow flow of air to the reactor cavity and prevent backflow	Provide circulation of air		Provide circulation of air
Description <u>of Component</u>	004 backflow damper, train B, NC	Fan, fan shaft, bearing, etc., 1- 1516-B7-001-000		Fan, fan shaft, bearing, etc., 1- 1516-B7-002-000
Item No.	ω	O		10

a. A - Safety injection for trains A and B start automatically and operate on safety injection only.

TABLE 6.2.5-3

DESIGN DATA FOR PRINCIPAL COMPONENTS OF POST-LOCA CONTAINMENT HYDROGEN PURGE SYSTEM

Containment Building Post-LOCA Purge Filter Exhaust Unit
Quantity 1
Capacity (ft³/min) 500
System Components
Charcoal Adsorber
Efficiency (%) 89.0^(a)
Face velocity (ft/min) 40
Residence time (s/2-in. thickness) 0.25
Nominal size (Tyler mesh) 8 x 16

HEPA Filters

Filter element Pleated fiberglass Size (in.) 24 x 24 x 12

Efficiency (%) 99.97% for 0.3-μm and larger

particulates

Moisture Eliminator

Separator element Fiberglass or galvanized steel Efficiency (%) 99% for 5 to 10 μ m droplets

Electric Heating Coil

Heater element 80% Ni, 20% Cr

Heating capacity (kW) 2.5

(a) Four-inch filter tested per Table 2 of Regulatory Guide 1.140. With bypass leakage of 1%, the charcoal adsorber section efficiency is 89%.

TABLE 6.2.5-4

DESIGN DATA FOR PRINCIPAL COMPONENTS OF POST-LOCA CAVITY PURGE SYSTEM

Fan Quantity Type Capacity (ft³/min) Static pressure (in. WG)	2 Centrifugal 300 30
Static pressure (in. WG)	30
Motor (hp)	5

TABLE 6.2.5-5

DESIGN DATA FOR PRINCIPAL COMPONENTS OF THE CONTAINMENT HYDROGEN MONITORING SYSTEM

Hydrogen Analyzer

Quantity 2 per unit

Type Thermal conductivity

Range 0 to 1 and 0 to 10 volume percent

Accuracy ±5 percent of full scale

Valves (isolation)

Quantity

Type 8 solenoid-operated globe valves and

10

2 check valves

Tubing Material Stainless steel (Class 2)

TABLE 6.2.5-6 (SHEET 1 OF 4)

PLANT PARAMETERS USED TO CALCULATE POST-ACCIDENT HYDROGEN PRODUCTION

Core thermal power (MWt) (a)	3565
Containment free volume (ft³)	2.75×10^6
Normal containment temperature (°F)	120
Weight of zirconium clad incore (lb)	45,914
Percent zirconium-water reaction (%)	1.5
Hydrogen recombiner flowrate (sf ³ /min)	100
Hydrogen recombiner efficiency (%)	95

Baseline Aluminum Inventory in Containment

Component	Weight (lb)	Surface (ft²)
Flux map drive system	183	48
Nuclear instrumentation system	244	57
Digital rod position indicators	199	241
Control rod drive mechanism (CRDM) connectors	129	68
Miscellaneous valves (nuclear steam supply system) (NSSS)	230	86
Radiation monitoring system	4	4
Containment fan cooler return bend assemblies	765	3000
Communication equipment	65	10
Miscellaneous valves (balance of plant (BOP))	11	2
Containment lighting fixture plugs	10	6
Contingency (NSSS)	_250_	85
Baseline Total	2,090	3,607

TABLE 6.2.5-6 (SHEET 2 OF 4)

Baseline Zinc Inventory in Containment

	<u>Type</u>	Weight (lb)	Surface (ft²)	
Snubbers	Zn ^(b)	11	555	
Integrated reactor vessel (RV) head/CRDM shroud	ZBP ^(c)	17,734	180,100	
Platform grating	GS ^(d)	12,453	99,625	
Pressurizer grating	GS	696	5,566	
Steam generator grating	GS	813	6,500	
Cables and related items	Zn	4,988	56,127	
Cable tray supports	Zn	1,710	13,680	
Inorganic zinc-based paint	ZBP	23,476	500,691	
Miscellaneous BOP items	ZBP	<u>36</u>	31_	
Baseline Total		61,917	862,875	

TABLE 6.2.5-6 (SHEET 3 OF 4)

Regulatory Guide 1.7 Hydrogen Production Calculational Assumptions

Core Cooling Solution Radiolysis

Sources:

	Percent of total halogens retained in the core	50
	Percent of total noble gases retained in the core	0
	Percent of other fission products retained in the core	99
En	ergy absorption by core cooling solution:	
	Percent of gamma energy absorbed by solution	10
	Percent of beta energy absorbed by solution	0
Ну	drogen production:	
	Molecules hydrogen produced per 100 eV energy absorbed by solution	0.50

TABLE 6.2.5-6 (SHEET 4 OF 4)

Sump Solution Radiolysis

Sources:	
Percent of total halogens released to sump solution	50
Percent of noble gases released to sump solution	0
Percent of other fission products released to sump solution	1
Energy absorption by sump solution:	
Percent of total energy (beta and gamma) which is absorbed by the sump solution	100
Hydrogen production:	
Molecules of hydrogen produced per 100 eV of energy absorbed by the sump solution	0.5
Long-Term Aluminum Corrosion Rate	
Mils per year	200

a. Hydrogen generation analysis for core thermal power of 3565 MWt is bounding for	
MUR power uprate of 3625.6 MWt (see figure 6.2.5-7)	
- p p	
b. Zn - zinc metal.	

c. ZBP - zinc-based paint.

d. GS - galvanized steel.

TABLE 6.2.5-7 CORE FISSION PRODUCT ENERGY AFTER 650 FULL-POWER DAYS

DELETED

TABLE 6.2.5-8 FISSION PRODUCT DECAY DEPOSITION IN SUMP SOLUTION

DELETED

TABLE 6.2.5-9

POST-ACCIDENT CONTAINMENT TEMPERATURE TRANSIENT FOR HYDROGEN GENERATION ANALYSIS

Time Interval(s)	Temperature(°F)
0 - 1.0	rapid increase from 120 to 259
1.0 - 4 E3	259
4 E3 - 6 E3	Ramp down from 250 to 240
6 E3 - 1 E4	Ramp down from 240 to 225
1 E4 - 2 E4	Ramp down from 225 to 205
2 E4 - 4 E4	Ramp down from 205 to 190
4 E4 - 6 E4	Ramp down from 190 to 175
6 E4 - 1 E5	Ramp down from 175 to 160
1 E5 - 1.00 E6	Ramp down from 160 to 131 ^(a)
> 1.00 E6	131

a. The long-term aluminum corrosion rate of 200 mils per year begins when the containment temperature drops below 154°F.

TABLE 6.2.5-10

CORROSION RATES USED IN THE POST-ACCIDENT CONTAINMENT HYDROGEN GENERATION ANALYSIS

Temperature <u>(°F)</u>	Aluminum Corrosion Rate (Ib-moles _{al} /ft²-h)	Zinc Corrosion Rate ^(a) (Ib-moles _{zn} /ft²-h)
100	1.19 E-5	7.53 E-9
140	-	7.16 E-8
154	1.19 E-5	-
160	1.49 E-5	2.21 E-7
180	3.15 E-5	6.80 E-7
200	6.65 E-5	2.10 E-6
220	1.40 E-4	6.47 E-6
240	2.97 E-4	1.99 E-5
260	6.27 E-4	6.15 E-5
280	1.32 E-3	1.90 E-4
300	2.80 E-3	5.84 E-4

a. Zinc corrosion rate also applies to zinc-based paint.

TABLE 6.2.6-1 (SHEET 1 OF 2)

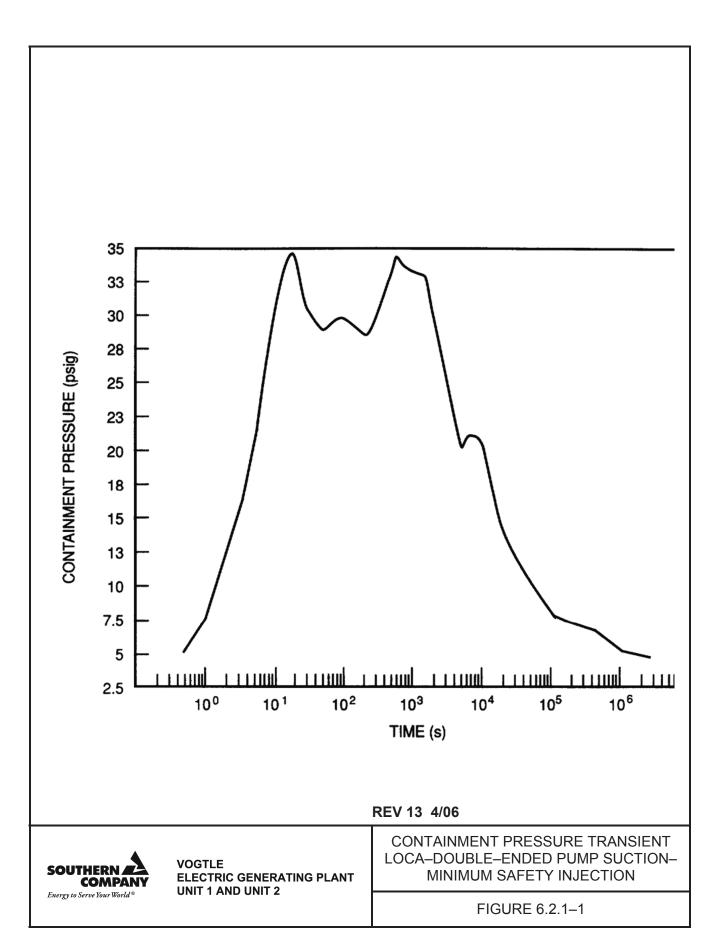
PENETRATIONS NOT VENTED TO CONTAINMENT OR DRAINED DURING TYPE A TESTING

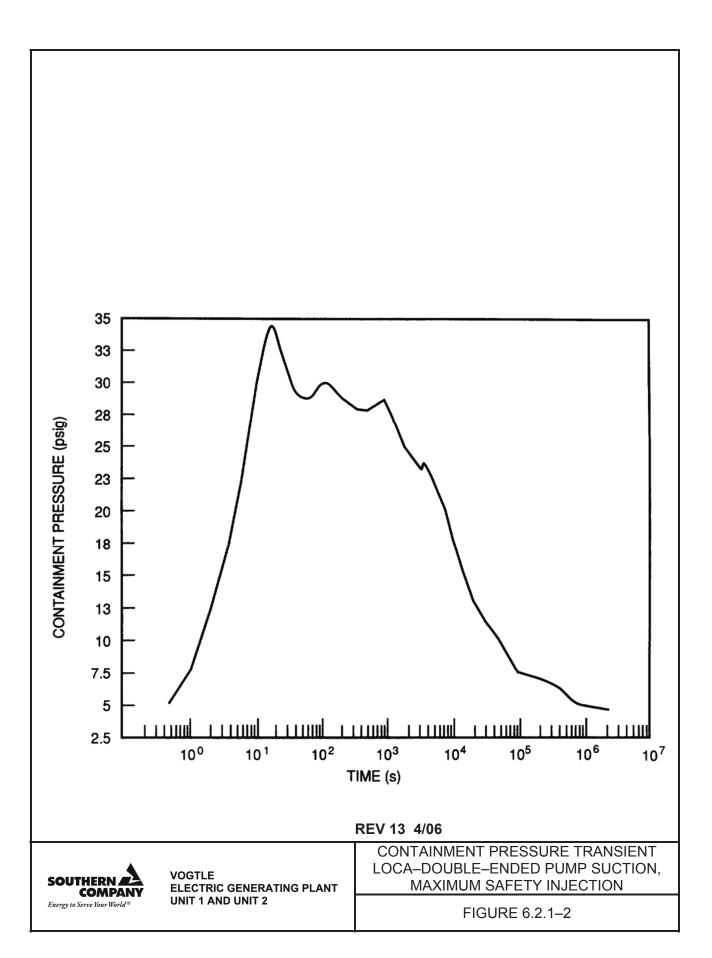
<u>Justification</u>	The steam generator secondary side is considered an extension of the containment. The system is not part of the reactor coolant pressure boundary and does not open directly to the containment atmosphere during normal and post-accident conditions. However, since the secondary side of the steam generator is a viable leak path, the main steam lines are vented outside of containment.	The system is normally filled with water from refueling water storage tank and operating under post-accident conditions.	The system is normally filled with water from the charging pump discharge and operating under post-accident conditions.	The system is normally filled with water from refueling water storage tank and operating under post-accident conditions. After an accident the static head between emergency sump and valve provides water seal to prevent leakage of containment atmosphere.	The system is normally filled with water from the refueling water storage tank and operating under post-accident conditions.	The system is closed outside containment and constructed to ASME III, Class 2, and Seismic Category 1 standards.
<u>Description^(a)</u>	Steam generator secondary side (main steam blowdown, sampling, feedwater, and auxiliary feedwater)	Safety injection line	Boron injection line (high head safety injection)	Residual heat removal and containment spray pump suction from containment emergency sump	Residual heat removal discharge to reactor coolant system	Residual heat removal suction from hot leg
Penetration <u>Number^(a)</u>	1-4, 7-10, 11B, 11C, 12B, 12C, 18-21, 101-104	30, 31, 33	32	36-39	56, 57, 58	59, 60

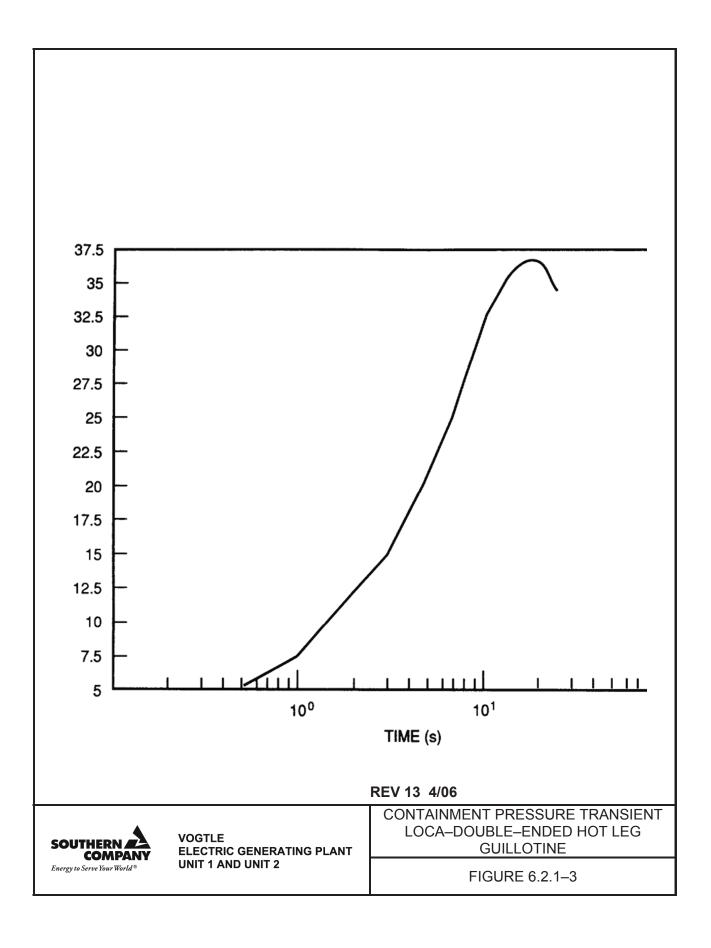
TABLE 6.2.6-1 (SHEET 2 OF 2)

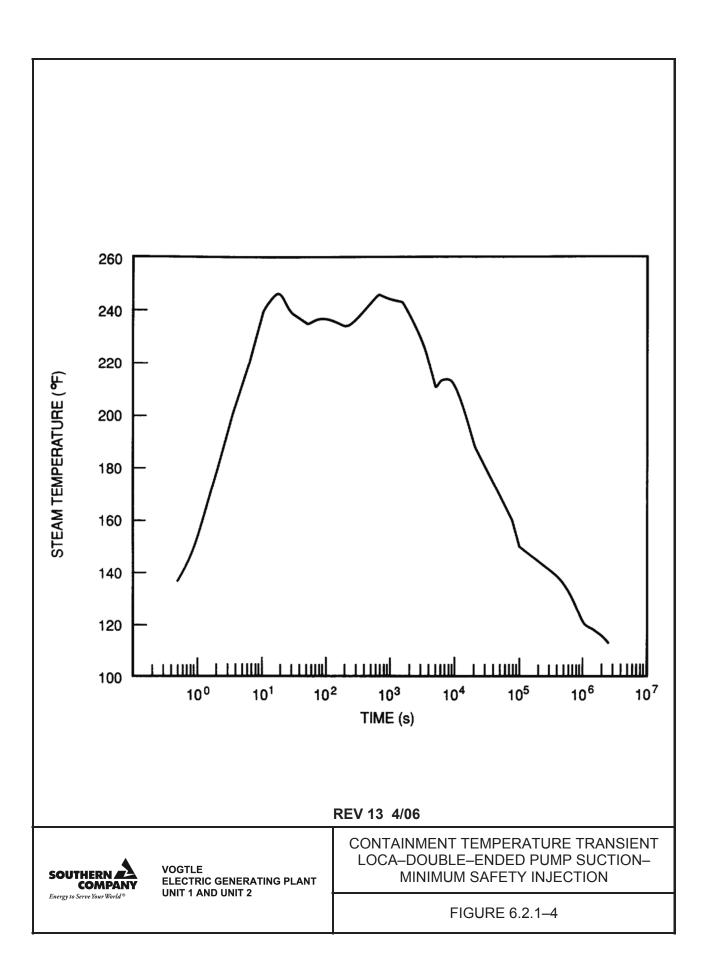
Justification	The system is a closed system inside containment per GDC 57 and thus not a potential atmospheric leak path. The system is normally filled with water and operating post-accident. The system may be operated to cool the containment atmosphere during the Type A tests.	The system is closed outside containment and constructed to ASME III, Class 2, and Seismic Category 1 standards. The system is filled with water during all modes of plant operation (normal and post-accident) by the charging pumps.	The system is filled with liquid and designed to satisfy the requirements of Regulatory Guide 1.141. Also the system is closed both inside and outside the containment.	The system is filled with liquid and designed to satisfy the requirements of Regulatory Guide 1.141. Also the system is closed both inside and outside.
Description	Nuclear service cooling water supply and return	RCP seal water supply	Containment pressure detectors	Reactor vessel water level instrumentation
Penetration <u>Number</u>	43-46, 91-98	51-54	13C, 67C, 69C, 70C, 71C, 85C	14A, 14B, 14C, 88A, 88B, 88C

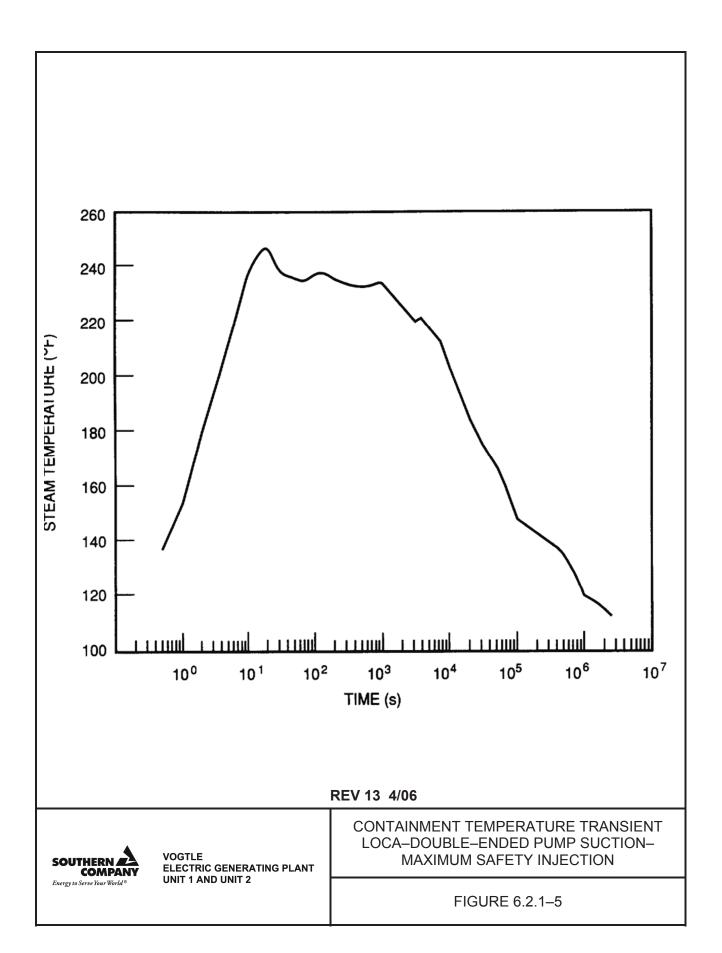
a. See table 6.2.4-1 for further description.

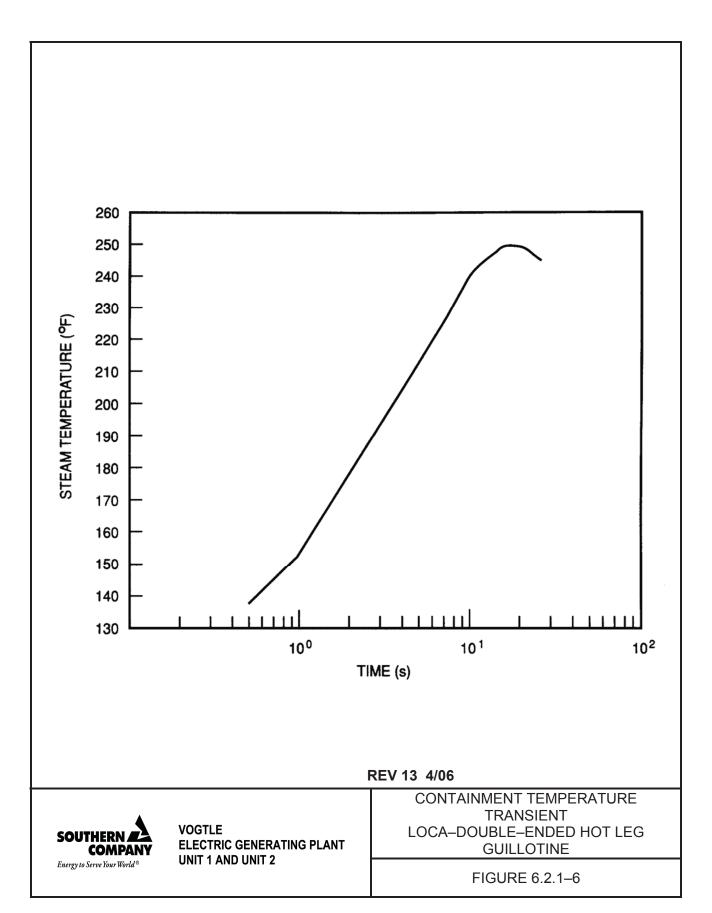


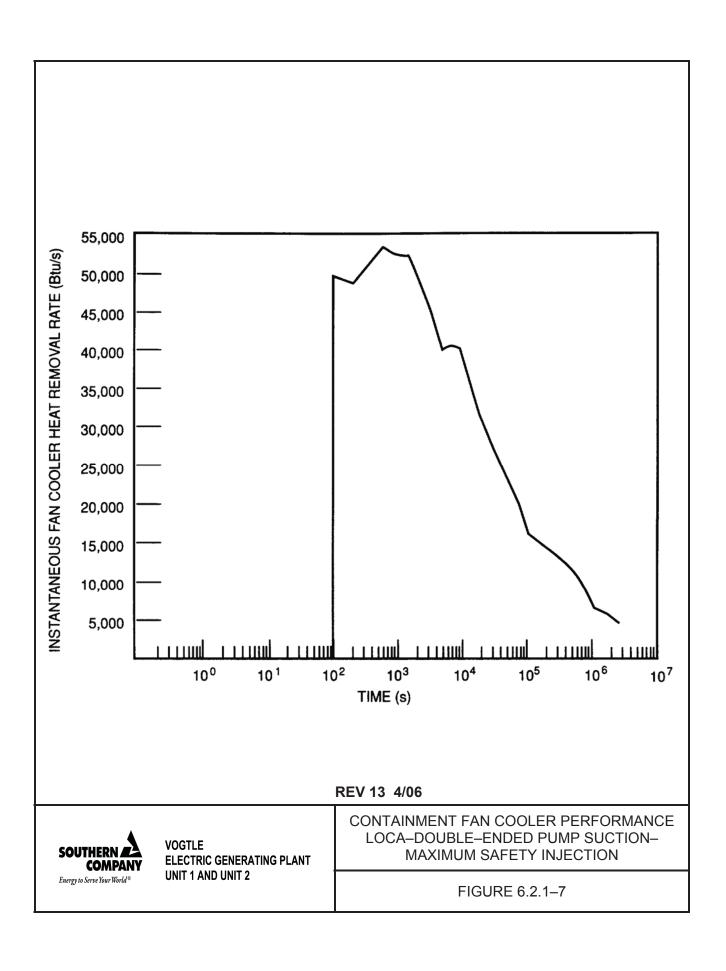












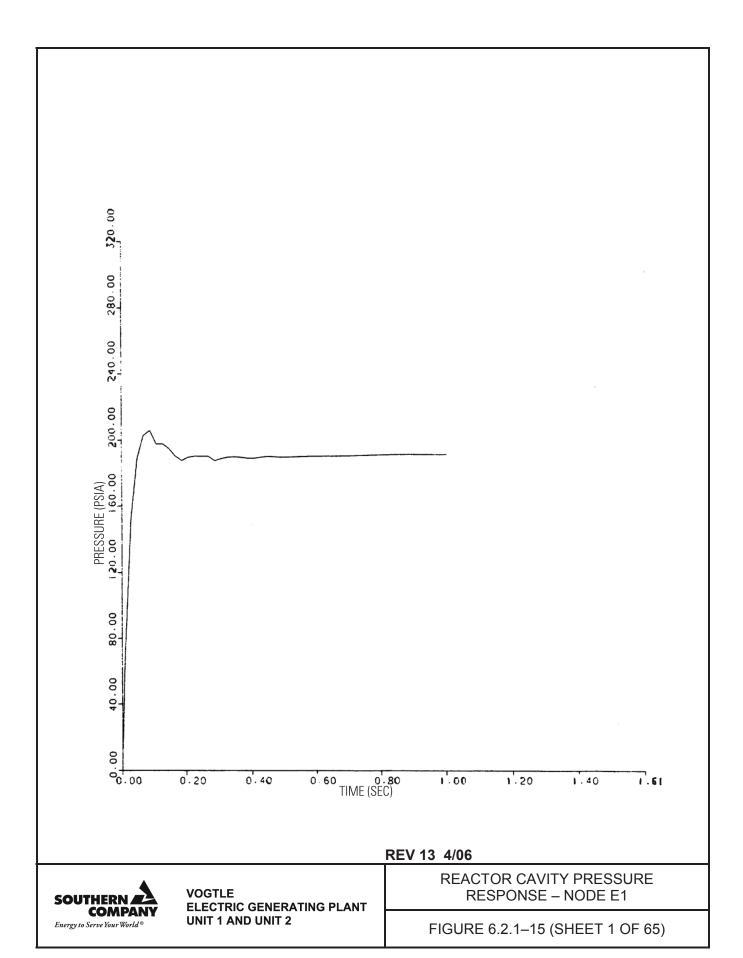
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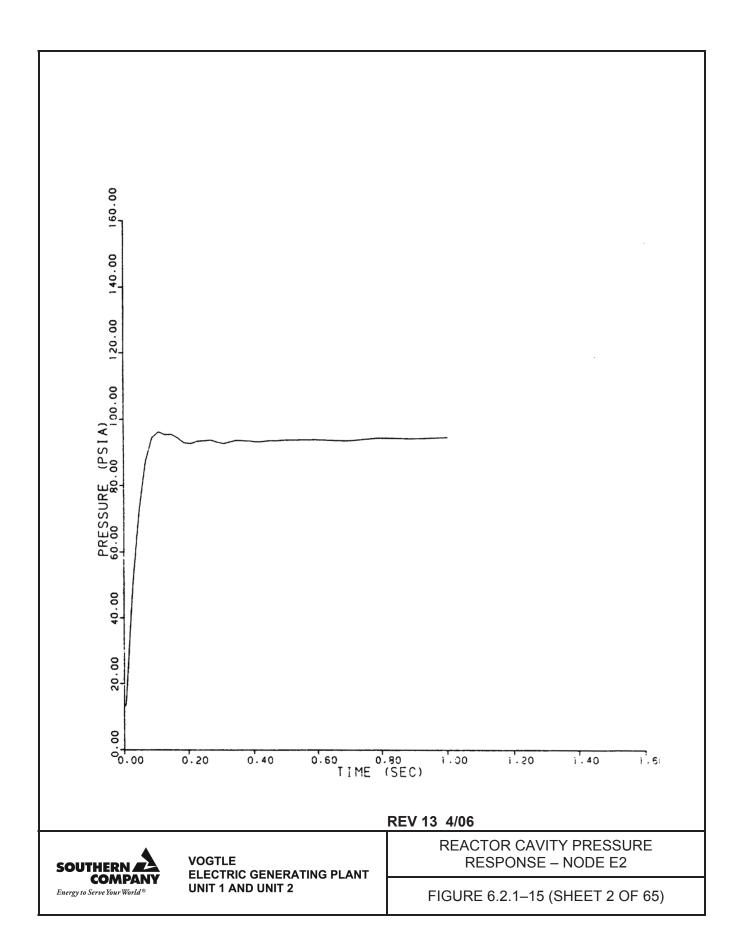
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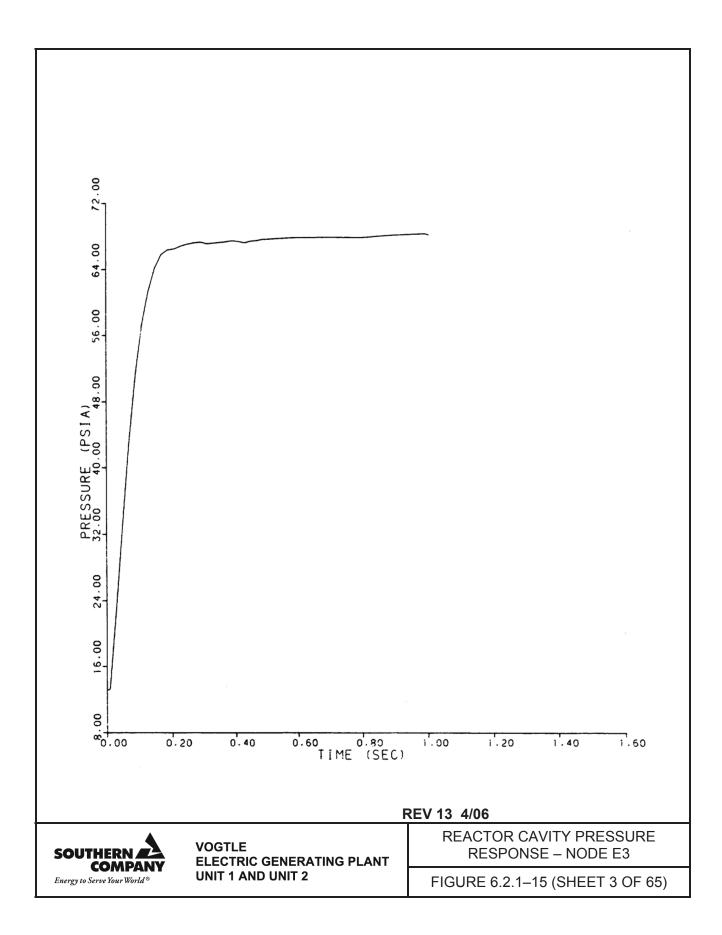


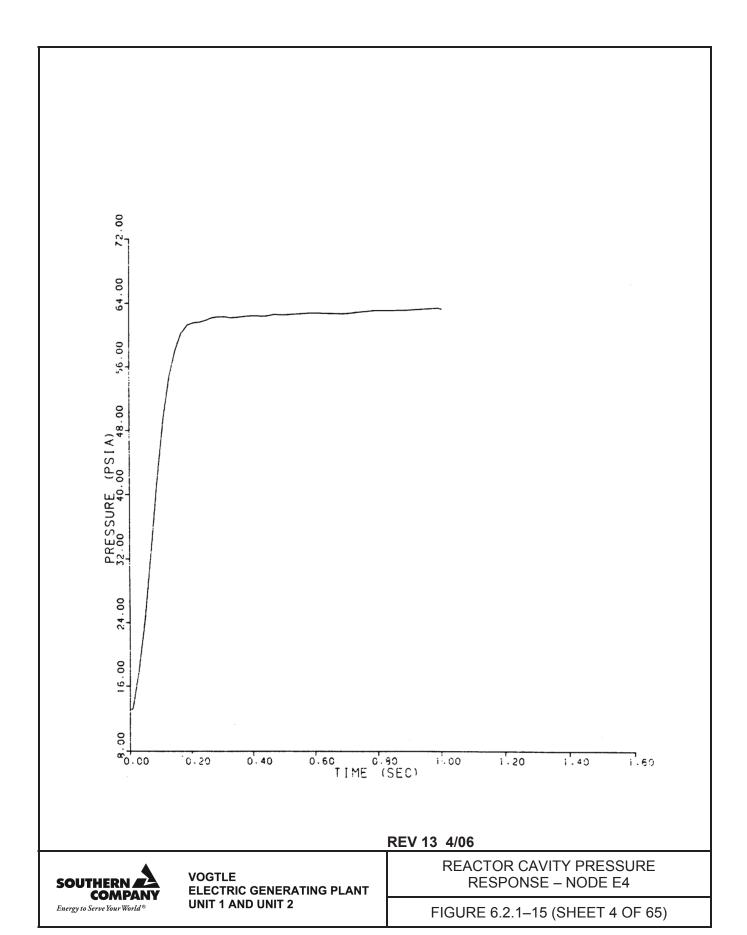
VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2

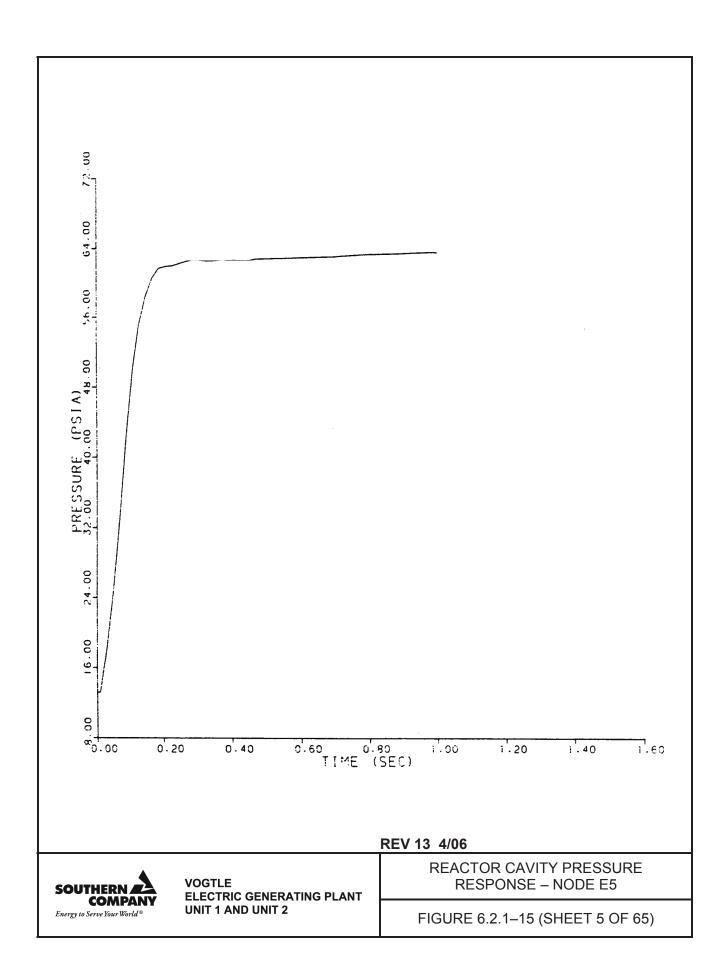
FIGURE 6.2.1–8 THROUGH FIGURE 6.2.1–14

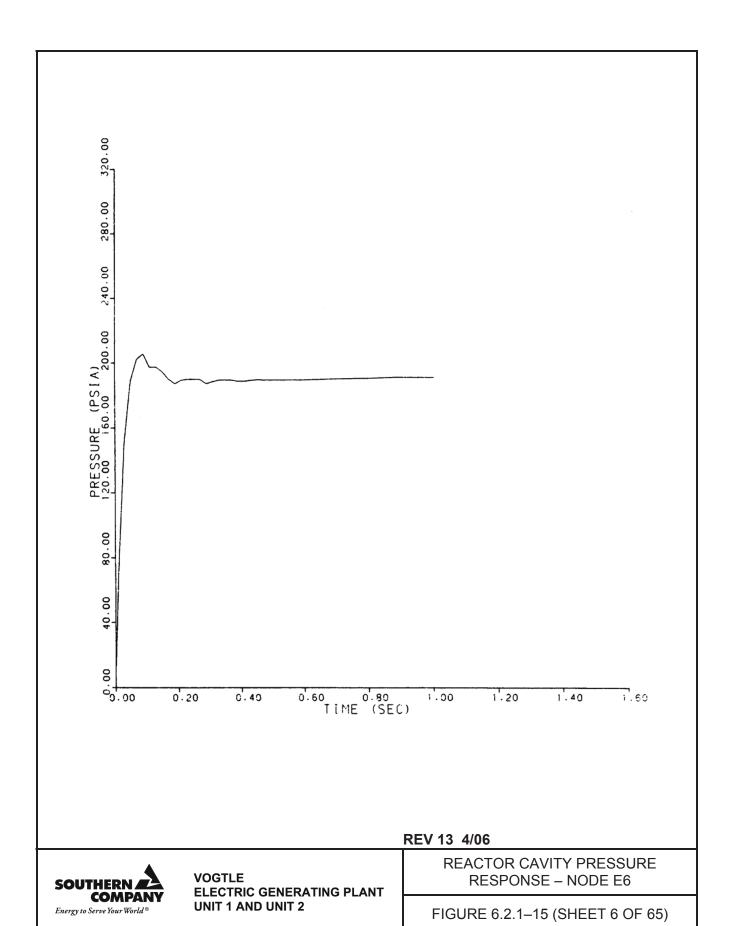


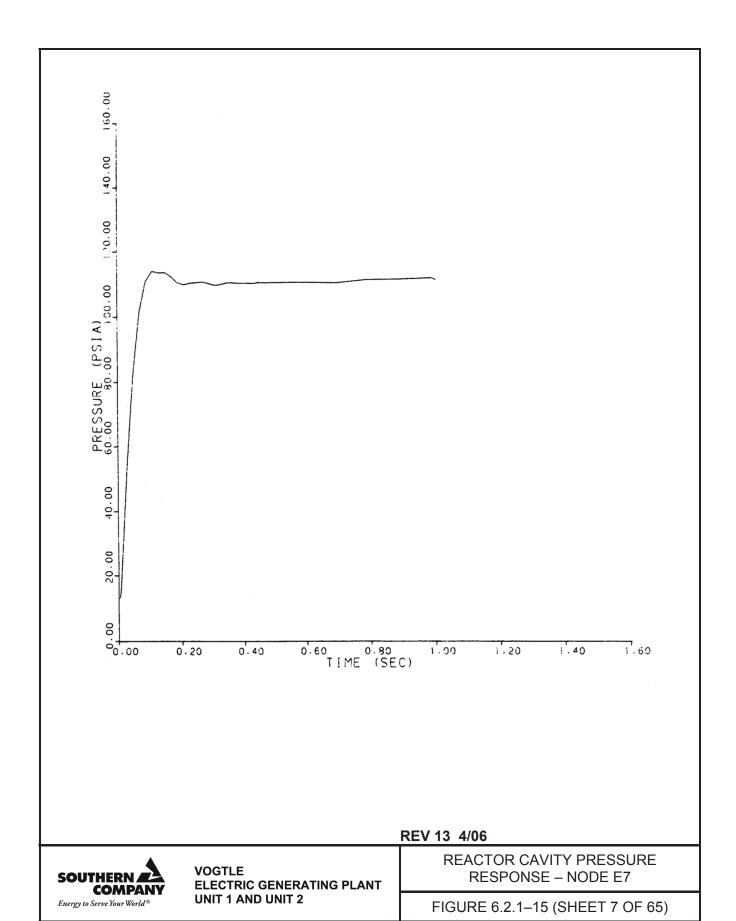


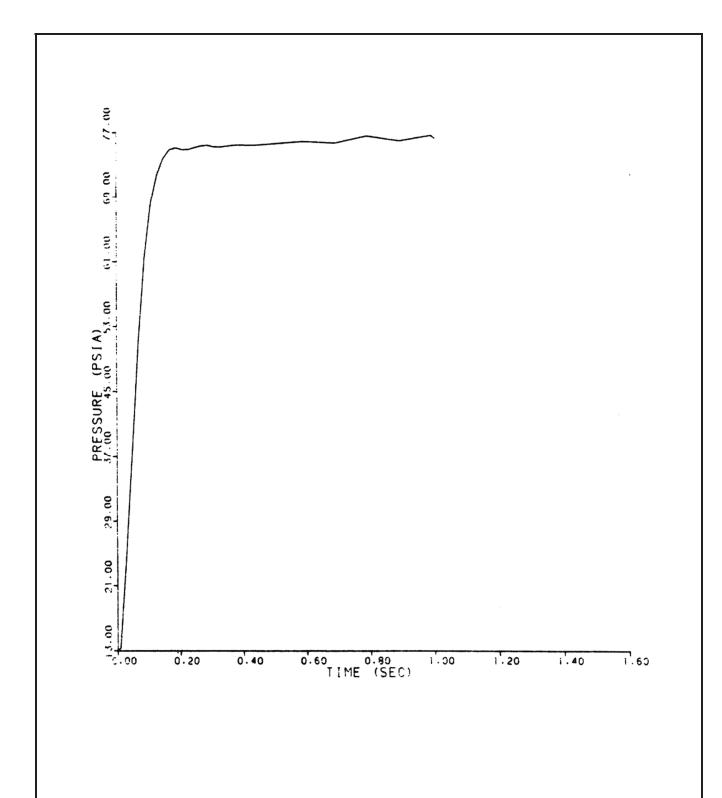














VOGTLE ELECTRIC GENERATING PLANT UNIT 1 AND UNIT 2 REACTOR CAVITY PRESSURE RESPONSE – NODE E8

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FIGURE 6.2.1-15 (SHEET 8 OF 65)

