

LTS PROGRAM
CONFIRMATORY ANALYSIS
OF SCNGS 1
PIPING AND PIPE SUPPORTS

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ABSTRACT

A series of confirmatory piping and pipe support analyses have been performed to ensure that the criteria and methodology employed by Southern California Edison Company (the licensee) in seismically upgrading the San Onofre Nuclear Generating Station, Unit 1 (SONGS 1), are acceptable. Four piping systems--SI-51, SI-04, SI-158, and a run of SI-51 tubing--were subjected to confirmatory finite element analysis. These analyses were performed to the requirements of the USNRC Systematic Evaluation Program (SEP) guidelines and current industry practice. In addition, the run of tubing was subjected to a confirmatory hand calculation. Supports for the SI-51 and SI-04 systems, a total of 50 supports, were also analyzed. Results of the confirmatory finite element analyses were compared with those of the licensee. Results were compared for the tubing between the finite element analysis and the hand calculation. The conclusion was drawn that the criteria and methodology employed by the licensee in analyzing the piping and pipe supports of SONGS 1 meet the requirements and are acceptable.

ACKNOWLEDGMENTS

This report is a product of the efforts of several people.

S. L. Morton wrote the first draft of the report and did much of the piping analysis. Much of the pipe support analysis was done by V. B. Call and C. Edgar. B. J. Quintana provided invaluable technical assistance.

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1. INTRODUCTION

A confirmatory analysis of the piping and supports at the San Onofre Nuclear Generating Station Unit 1 (SONGS 1) was requested by the Nuclear Regulatory Commission (NRC). This analysis was performed in an effort to verify a portion of the SONGS 1 Long Term Service Seismic Reevaluation Program,^a (Reference 1) specifically, the analysis techniques utilized by the SONGS 1 piping and support analysts for the LTS Program. Three Safety Injection System (SIS) piping models and one SIS tubing model were selected to represent the wide variety of pipe sizes incorporated in the plant. The four pipe sections chosen for detailed examination were SI-51, SI-04, SI-158, and SI-51 (tubing). Emphasis was placed on modeling techniques, accuracy, loading conditions, and pipe and support stresses. This report summarizes the confirmatory analyses performed on the four models and associated supports. It also provides a comparison of results for the analyses done by EG&G and by the licensee, Southern California Edison Co.

a. Hereafter referred to as the LTS Program.

2. PIPING SYSTEM DESCRIPTION

Information pertaining to the details of these piping systems was provided by the licensee. Much of it was verified by independent line walkdowns by the author. The information provided contained the latest as-built pipe dimensions, pipe specifications, valve weights, response spectra, seismic and thermal anchor movements, detailed pipe support drawings, and other miscellaneous data required to perform the independent piping analyses.

SIS problem SI-51 includes the piping between feedwater pumps FWS-G-3A and FWS-G-3B and the six-inch branch piping from this line to the containment sphere. Figures 1 through 6 are computer model plots defining problem SI-51. The non-safety related piping included in this analysis, shown in these computer plots, was provided to include its effects on the safety related piping. SIS problem SI-04 includes the piping between pump SIS-G-50A and feedwater pump FWS-G-3A. Computer model plots, Figures 7 through 9, describe the problem in more detail. SIS problem SI-153, seen in Figures 10 through 13, is a small model which includes six-inch piping from the containment sphere penetration B-1B to the cold leg of reactor coolant loop C. Finally, tubing problem SI-51 is defined in Figures 14 and 15. The tubing runs from flow transmitter SIS-FT-912 to valve 870E-3/4"-T57 (on the SI-51 piping).

Tables 1 through 4 list the significant pipe data and material specifications utilized in the analyses described in this summary. Design and service conditions are also included in these tables.

Various service levels were analyzed. This included deadweight, pressure, thermal expansion, thermal anchor movements, and the inertial and anchor motions due to the 0.67 g modified Housner response spectra earthquake. These analyses were performed utilizing the ASME Code Class 2 rules (Reference 2) as specified by the SEP Guidelines (Reference 3) and compared to the stresses calculated by the licensee.

3. PIPING SYSTEM SEISMIC ANALYSES

All four finite element analyses were performed using the program NUPIPE-II, a proprietary program developed by Quadrex Corporation. NUPIPE-II capabilities are briefly described in Appendix A. Idaho National Engineering Laboratory (INEL) program module V4AGINL was utilized for these analyses.

The piping system data, drawings, and response spectra necessary for these analyses were provided by the licensee. The following assumptions based on experience and engineering judgment were employed where information was missing or incomplete:

1. All components meet the dimensional requirements of the standards listed in Table NC-3132-1 of the ASME Code.
2. The requirements of the ASME Code, Section III, Subarticle NC-3640 are satisfied. This subarticle deals with ensuring adequate wall thickness of the components for the Design Pressure at the Design Temperature. Since the change in design requirements involves the seismic load case only and pressure stress has been included in the seismic analyses, an explicit check for adequate wall thickness was deemed unnecessary.
3. The only thermal mode assumed was the normal operating mode.

A summary of weight and center of gravity information for all valves on the four safety injection piping models is contained in Table 5. This table also includes the face-to-face dimension for each of the valves.

The four models were analyzed for weight, thermal expansion plus thermal anchor movements, earthquake, and seismic anchor motion (SAM) loads. All were analyzed in accordance with the SEP Guidelines (Reference 3). Details of the analyses not specified in this document were in accord with standard EG&G piping analysis practice. These details were

at a level not controlled explicitly by the regulatory process. There is one exception to this. EG&G SAM load generation methodology involves envelopment of structural motion displacements in the area of the support to obtain SAM values. These are then applied individually, with the results combined by square root of the sum of the squares (SRSS) combination. Preliminary result comparisons showed significant differences between EG&G and utility SAM stresses. This initiated an investigation to identify the source of the differences. A different SAM load generation methodology is used by the licensee. SAM values are obtained by interpolation between appropriate structural motion displacements, with either algebraic or absolute value combination of loadings depending on the location and direction of the supports. The basis for this methodology was evaluated, and it was found acceptable. There are a sufficient number of points on the structural steel model with associated SAM data to allow interpolation for support point attachment SAMs. Algebraic combination of specified directions and locations is based on a study of the motion of the buildings which concluded that these particular directions and locations move in phase. Absolute summation of the remainder is conservative. The licensee's methodology was adopted in the EG&G analyses in order to prevent the potential masking of other sources of discrepancy.

An additional analysis was done for the SI-51 tubing model. The second analysis of the SI-51 tubing was done according to the alternate calculation methodology allowed by the licensee's small-bore piping criteria. This methodology was found to predominate in the small-bore piping calculations audited. It consists of an equivalent static analysis based on a simply supported beam calculation.

PVRC damping spectra obtained from the licensee's structural analyses were appropriately enveloped for each particular piping model and used for the earthquake load case in these analyses. These enveloped response spectra curves are shown in Figures 16 through 27. The spectra were applied in all three directions simultaneously and the results were combined using the square-root sum of the squares (SRSS) method. All seismic anchor movements were applied and combined using the licensee's methodology, as discussed above.

Microfiche copies of the finite element analyses are contained in
- Appendix B. The small-bore piping hand calculation is contained in
Appendix C.

4. PIPING ANALYSIS RESULT COMPARISONS

The piping finite element analyses were performed per the requirements of the SEP guidelines and the ASME Class 2 equations specified there. The results obtained from the loadings previously described were used to calculate the piping stresses. The small-bore piping hand calculation was performed using the methodology found to predominate in the audits of small-bore piping calculations. Allowable stresses, per the SEP guidelines, were $2.4S_h$ for the equivalent of Class 2 piping and $1.8S_h$ for the equivalent of Class 1 piping. Except for a small length of pipe near the reactor coolant loop, the piping analyzed is the equivalent of Class 2 piping.

Although calculated stresses were compared to allowable stresses, the focus of this work was a comparison of stresses as calculated by EG&G and the licensee to ensure that the stress calculation criteria and methodology employed by the licensee is acceptable. Some differences in methodology were found. Different techniques were initially used to combine seismic anchor motion (SAM) loads. This is discussed in detail in Section 3 of this report. The licensee's technique was found acceptable and adopted in the EG&G analyses to prevent the differences in this methodology from masking any other differences which may have existed. Other differences in methodology were found. This includes differences in valve and reducer modeling techniques and anchor point stiffness calculations. These are discussed in the following paragraphs.

As is common industry practice, all modes of vibration with natural frequencies under 33 Hz were included in the dynamic analyses. The effects of the higher frequency modes, the "missing mass effect" were automatically included in the results by the NUPIPE-II program. Tables 6 through 9 show the first ten modes of vibration for each of the piping models. The SI-51 model was calculated to have 52 natural frequencies below 33 Hz with rod hangers modeled rigidly and 50 without rod hangers. The SI-04 model displayed 42 natural frequencies under 33 Hz with rigid rod hangers and

43 without rod hangers. The SI-158 model, which has no rod hangers, had 13 natural frequencies under 33 Hz. The SI-51 tubing model exhibited 17 natural frequencies below 33 Hz.

Comparisons of stress results are presented in Tables 10 through 13. The first column of each table identifies the EG&G node point and the licensee's data point at which stresses are compared. The EG&G node point (the first number) can be used to locate the point under consideration in the plots of the models (Figures 1 through 15). The next two columns identify the type of component being analyzed and its associated stress intensification factor (SIF). The remaining six columns are divided into two sets, one for the seismic inertia plus normal operating load (SI) stress, and one for the seismic anchor motion load (SAM) stress. This separation allowed a clear identification of the source of any differences in stress results. There are three columns for each type of stress. The first is for stresses from the EG&G analysis, the second for the licensee's stresses, and the third for the percent difference between the two sets of results, with the average value used listed in the percentage column header. This type of comparison was chosen because it emphasized the most significant differences, those in the areas of maximum stress, and because it allows a comparison of results among the models.

Stress results comparisons were limited to points in the models with SIF greater than 1.0, and at support points in the straight runs of piping, since these represent local stress maxima. Due to the large size of the SI-51 model, stress results were compared only in areas of maximum stress, which were found to be in the same areas of the model in both sets of results. These areas were found in the vicinity of both feedwater pump nozzles and in the vicinity of the branch point for the west side 6-in. line running to containment. Stress result comparisons were presented for all of the SI-04 model except for nodes 5 through 60 and those nodes associated with boundary condition piping (non-safety related), where the comparison yielded identical trends to those presented. All stress result comparisons are presented for the SI-158 model.

Comparison of stress results for the SI-51 analyses shows close agreement (see Table 10). The EG&G stresses are based on as-built data and standard practice at EG&G with the following exceptions. The rod hanger support at node 145 (Figure 3) was given full credit for vertical support. Although not in agreement with the as-built condition, this reflects the licensee's decision to upgrade the rod hanger to a full vertical support (typically done by adding a bumper near the rod hanger to restrain uplift of the piping). This modification to EG&G's SI-51 model was done to obtain an identical support configuration with the licensee's model. A two element reducer model was used in the EG&G analysis, each element having the section properties of the attached piping. This is in contrast with standard EG&G practice, which is to use a single element with the section properties of the smaller of the attached pipes, and to increase the stress on the large end of the reducer to reflect the larger pressure stress there. This practice is known to be overly conservative, and the refined two element model is used at EG&G in cases where the standard model indicates an overstress.

There were also instances of EG&G common practice different from the licensee's which were not changed. In EG&G common practice, anchor stiffness is based on the associated pipe section properties, while the licensee uses a single fixed value. These are the default procedures found in the two computer analysis codes used. EG&G valve element stiffnesses are based on a factored stiffness for the equivalent piping element, while the licensee's are based on the associated pipe element with a doubled wall thickness. Valve masses are distributed differently, although the center of gravity of the valve and total mass are the same. EG&G uses a single mass for both the valve body and the operator; the licensee uses a separate mass for each. Since the valve models are included only to simulate the effect of the valves on the piping, these differences are negligible. Centerline offsets for axial supports are modeled by EG&G but not by the licensee.

Differences in common practice between EG&G and the licensee had no significant effect on the stress results, as indicated in Table 10. Extremely close agreement was obtained, with the maximum difference appearing in the area of tee components. These differences are attributed to the accumulation of the differences in stiffness along each portion of the piping that terminates at the tee.

The results of the SI-04 analyses did not compare as favorably as those for the SI-51 analyses (see Table 11). The EG&G model was strictly as-built. This resulted in two differences between EG&G's and the licensee's models. The vertical support at node 265 (Figure 9) in the EG&G model was located at the position corresponding to node 260 in the licensee's model. The reducer between nodes 335 and 340 of the EG&G model was modeled as 14 in. long in the EG&G model, and 7 in. long in the licensee's model. In both cases, the EG&G model was closer to the as-built configuration, but both variations in the licensee's models were within IE Bulletin 79-14 tolerances. The same differences in common practice found in the SI-51 models were also found. Reducer model refinements made in EG&G's SI-51 analysis were not made in the SI-04 analysis.

Since the differences in stress results for the SI-04 analyses were larger than those for the SI-51 analyses, an attempt was made to isolate the cause. The EG&G SI-04 model was modified to duplicate the licensee's support configuration, valve and reducer models, and anchor stiffnesses. Stress results from analysis of the modified model were compared with no improvement seen. The location of stress variation was different, but the degree and number of variations were the same. The models were closely compared and found to be as near to identical as could be obtained using different analysis computer codes.

The source of differences in stress results was found in a contrast between the SI and SAM stress comparisons for the SI-04 analyses. The agreement in stresses is much better for the SAM load. This indicates that the discrepancies affect the dynamic analyses more strongly than the static analyses. Differences in SAM stress greater than 10% occur only in

reducers and elbows. This is not anomalous for the reducers, which are modeled differently, but it indicates that elbow element stiffness formulations differ slightly. The effect of this small variation is negligible for the static (SAM) analyses, but can be significant in the dynamic (SI) stress results. Such variations would have little effect on the calculation of the lower frequency modes of vibration, which are more accurately defined than the higher frequency modes in dynamic analysis. This characteristic makes the effect of minor variations in elbow stiffness on stress results dependent on the frequency content of the response spectra used in the dynamic analysis. Response spectra with relatively more excitation in the low frequencies, such as the SI-51 spectra, generate resultant stresses with larger contribution from the low frequency modes. The effects of the elbow stiffness variation are attenuated, and the stresses compare closely. The SI-04 spectra, having relatively less low frequency excitation, generate resultant stresses with relatively more contribution from the higher frequency modes. The effect of the elbow stiffness variation is emphasized, and the stress comparison is less favorable.

The existence of small variations in elbow stiffness formulation is not particularly harmful in seismic analysis, because the effects of the variation are concentrated in the calculation of the higher frequency response. Seismic events are essentially low frequency events, a result of the fact that high frequency motion is rapidly attenuated in transmission through the ground. Large seismic stresses are therefore associated with low frequency response, where the variation in elbow stiffness is not significant. This can be seen in a comparison between the maximum stresses calculated in the SI-51 and SI-04 analyses. The average maximum SI-51 analysis stress, which is in the range of the SEP guidelines allowable stress, is 2.6 times larger than the SI-04 average maximum stress.

A comparison of stresses for the SI-158 analyses are presented in Table 12. Good agreement is indicated. These results compliment the conclusions drawn in the discussion of the SI-51 and SI-04 results.

For an overall comparison, the differences in stress for all entries of Tables 10 through 13 were averaged for the seismic inertia and SAM stresses. On the average, EG&G seismic inertia stresses were 1.0 ksi higher, and the SAM stresses differed by less than 0.1 ksi. The difference in seismic inertia stress can be attributed to differences in spectral definition. EG&G spectra were limited to 30 points, while the licensee's spectra contained 160 points. In performing the reduction, envelopment criteria were imposed, which would result in a marginally higher stress calculated using the reduced spectra.

A comparison between finite element calculated stresses and allowable stresses showed the EG&G analyses and the licensee's analyses to be in exact agreement in terms of a conclusion of adequacy. SI-04 and SI-158 are acceptable with good margin. SI-51 had two components with stresses that exceeded allowable stresses. Both were reducers located on the discharge side of the feedwater pumps. This indicates that the licensee's criteria and methodology produce comparable results to those of the SEP guidelines. Supports have been added to the SI-51 piping to reduce stresses to acceptable levels.

A comparison between stresses calculated for the SI-51 tubing model using finite element analysis and hand calculations showed the hand calculation to be conservative. The maximum finite element analysis stress was 14.5 ksi, versus a hand calculated maximum stress of 71.8 ksi. According to the hand calculation, the tubing is severely overloaded. According to the finite element analysis (which is more accurate, and hence controls the conclusion concerning adequacy of the tubing), the tubing is acceptable to the SEP guideline limits with large margin.

Based on the detailed comparison of the results of three piping system finite element analyses, the criteria and methodology used by the licensee in analyzing piping were found to be in keeping with the SEP guidelines and current industry practice and are acceptable. The comparison of the confirmatory finite element analysis for the tubing with the confirmatory

hand calculation showed the hand calculation to be conservative and acceptable.

5. PIPE SUPPORT ANALYSIS RESULT COMPARISONS

Pipe support analyses were performed per the requirements of the SEP guidelines and current industry practice. Hand calculations were predominately used. Nominal stresses were calculated based on the loads supplied by the confirmatory piping analyses and nominal member section properties. Loads on catalog components were compared to vendor supplied allowable loads. Detailed finite element stress analyses were used in cases not amenable to hand calculation. Weight and thermal loads were considered for normal operating loads. Thermal loads were included only if they increased the resultant stress. Seismic inertia and seismic anchor motion loads were combined by the square root of the sum of the squares method, and then added and subtracted to the normal operating loads to obtain the range of loads used in calculating the seismic stress. In cases where the support geometry or loading was too complex for hand analysis, detailed finite element stress analyses were performed. The resulting stresses were compared to allowable stresses defined in the SEP guidelines.

All supports on safety related piping of the SI-51 and SI-04 models were analyzed. Since this led to the analysis of some 50 supports, analyses of the SI-04 supports were not performed. The analyses are contained in Appendix D. The supports for the SI-51 and SI-158 piping were then ranked according to decreasing stress ratio. The stress ratio for a support is defined to be the maximum ratio of calculated to allowable stress for all components of the support. Hence the support with the highest stress ratio is the most highly loaded relative to its allowable load. The licensee was requested to similarly rank the supports of both systems and to identify the supports with the largest stress ratio. For each piping system, the five supports with the largest stress ratio according to the EG&G analyses and the five according to the licensee's analyses were merged into a list. The lists were reviewed and found to contain a representative sample of the types of components found in pipe supports. The calculations supporting the stress ratios appearing on both lists were then subjected to a detailed comparison. The purpose of the comparison was to determine if the criteria and methodology applied by the licensee were acceptable.

Results of the comparison between support calculations are presented for the SI-51 piping supports in Table 13, and for the SI-158 supports in Table 14. Several differences in stress ratio were found, as indicated and discussed in the tables. No application of unacceptable criteria or methodology was found. Supports identified as overloaded by the EG&G analysis had either been forwarded to the support design group for modification, or shown to be acceptable by analysis of an improved support configuration. As noted in Section 4 of this report, supports were added to the SI-51 piping to reduce loads on the feedwater pump nozzles. Loads used in the licensee's support analyses were taken from the analysis of the piping with the improved support configuration. In some cases, the presence of added supports reduced loads on nearby supports, so that some of the supports identified as unacceptable in the EG&G analysis were shown acceptable under the reduced loading in the licensee's analysis.

Based on detailed comparison of the results of 14 supporting analyses, the criteria and methodology used by the licensee in analyzing pipe supports were shown to be in keeping with the SEP guidelines and current industry practice and are acceptable.

6. CONCLUSION

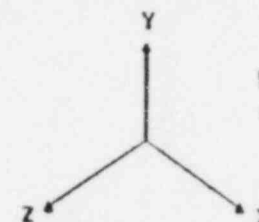
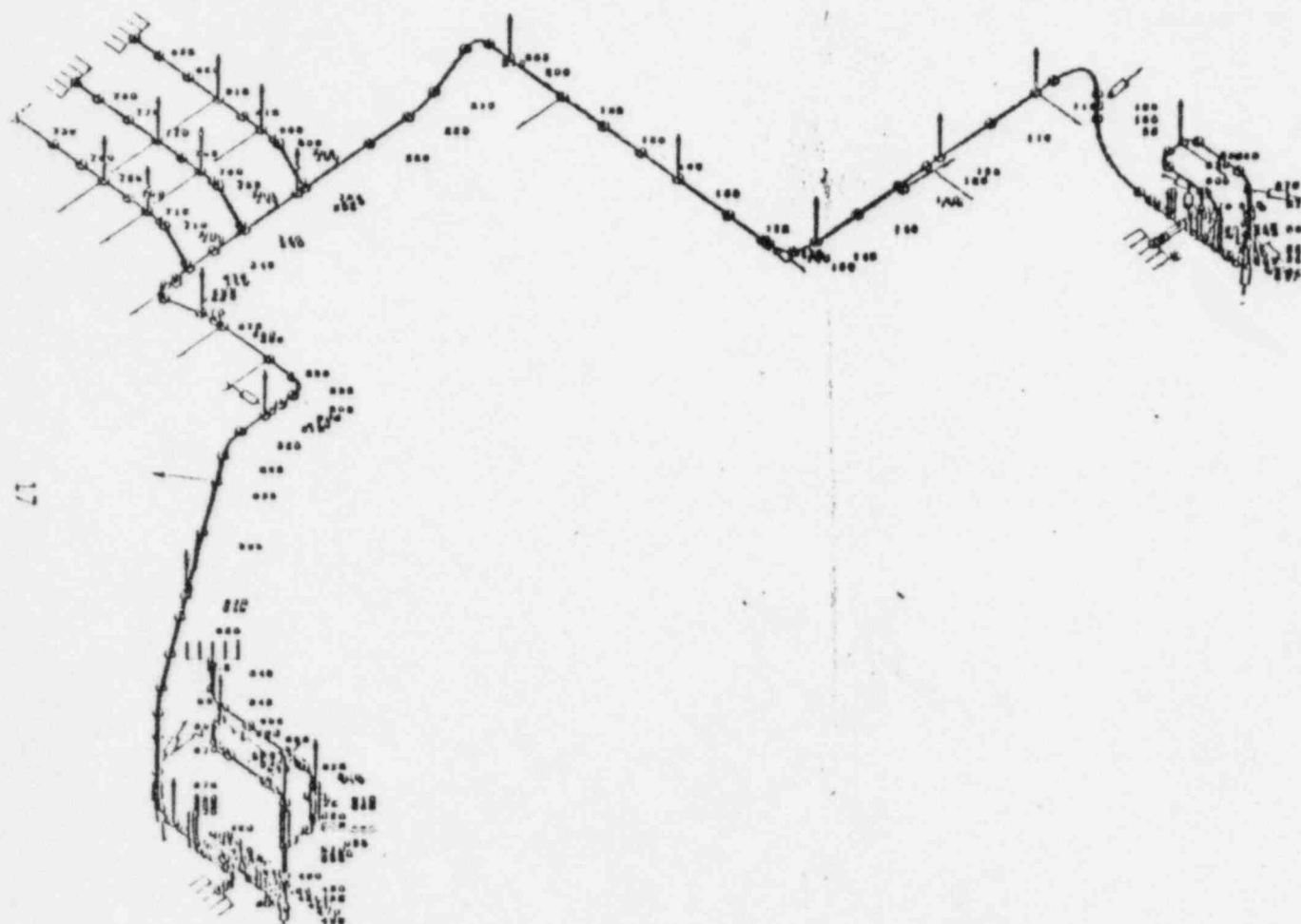
Four confirmatory finite element analyses and one confirmatory hand calculation were performed for piping systems. Pipe supports of two of the systems were subjected to confirmatory analysis. Results of the confirmatory analyses were compared to corresponding results of the licensee's analyses. The confirmatory hand calculation results were compared to the results of a confirmatory finite element analysis. Based on the comparisons, the licensee's criteria and methodologies for analyzing piping and pipe supports were found in keeping with the SEP guidelines and current industry practice and are acceptable.

7. REFERENCES

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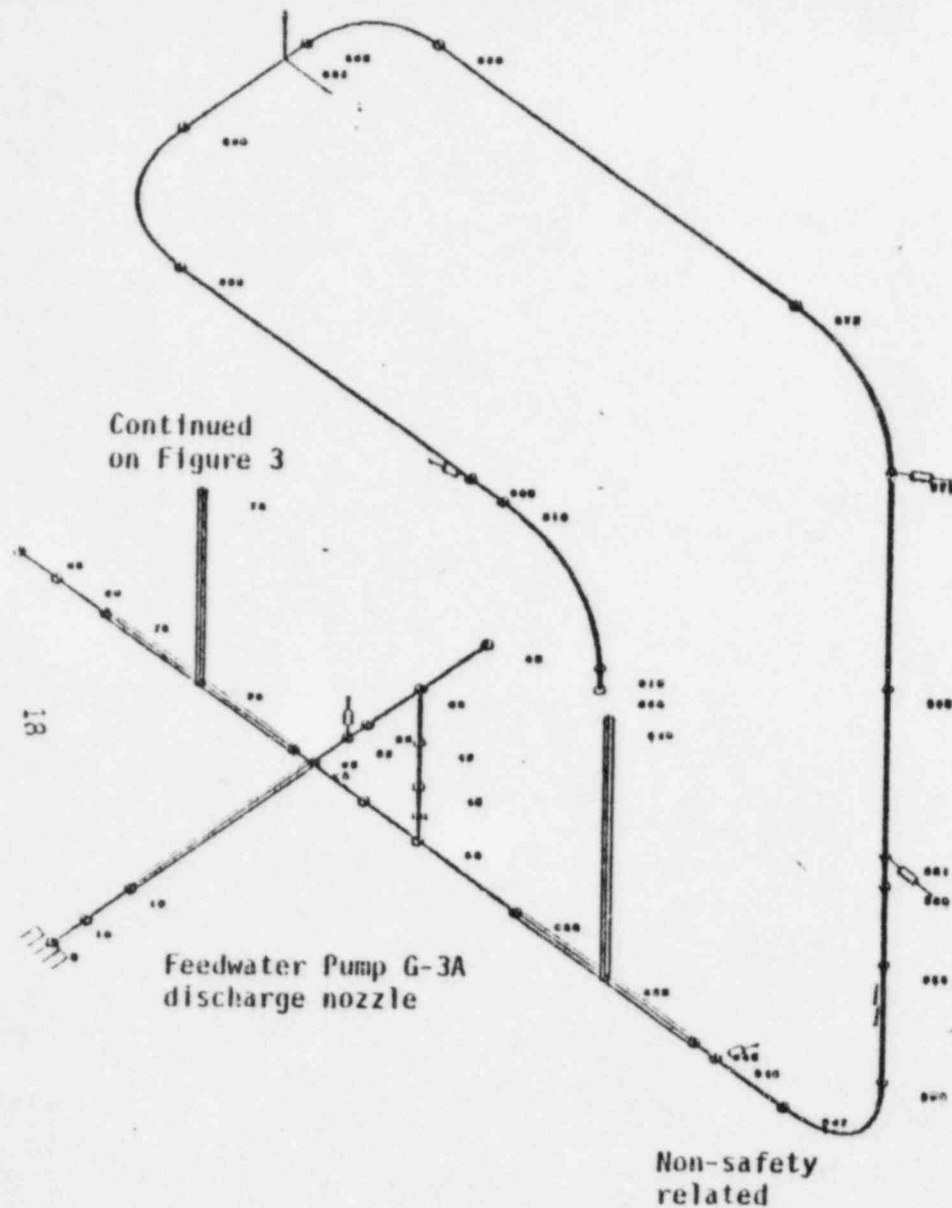
==LEGEND==

- / - NODE LOCATION
- ⊙ - HANGPOINT LOCATION
- ~ - SPRING HANGER
- - SNUDDER
- |— - RIGID SUPPORT
- |— - ANCHOR
- * - ELASTIC JOINT
- |— - FLEXIBLE ANCHOR
- |— - VALVE



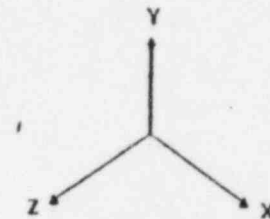
ROTATION ABOUT Y-AXIS * 45.4 DEG.
 X-Z PLANE TILT * 45 DEG.

Figure 1. Isometric of the SI-51 computer model.



==LEGEND==

- / - NODE LOCATION
- - MASSPOINT LOCATION
- ~ - SPRING HANGER
- - SHOCKER
- ⌋ - RIGID SUPPORT
- ≡ - ANCHOR
- * - ELASTIC JOINT
- ≡ - FLEXIBLE ANCHOR
- ⌋ - VALVE

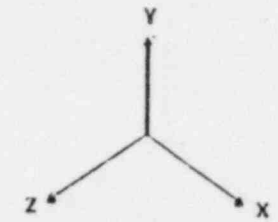
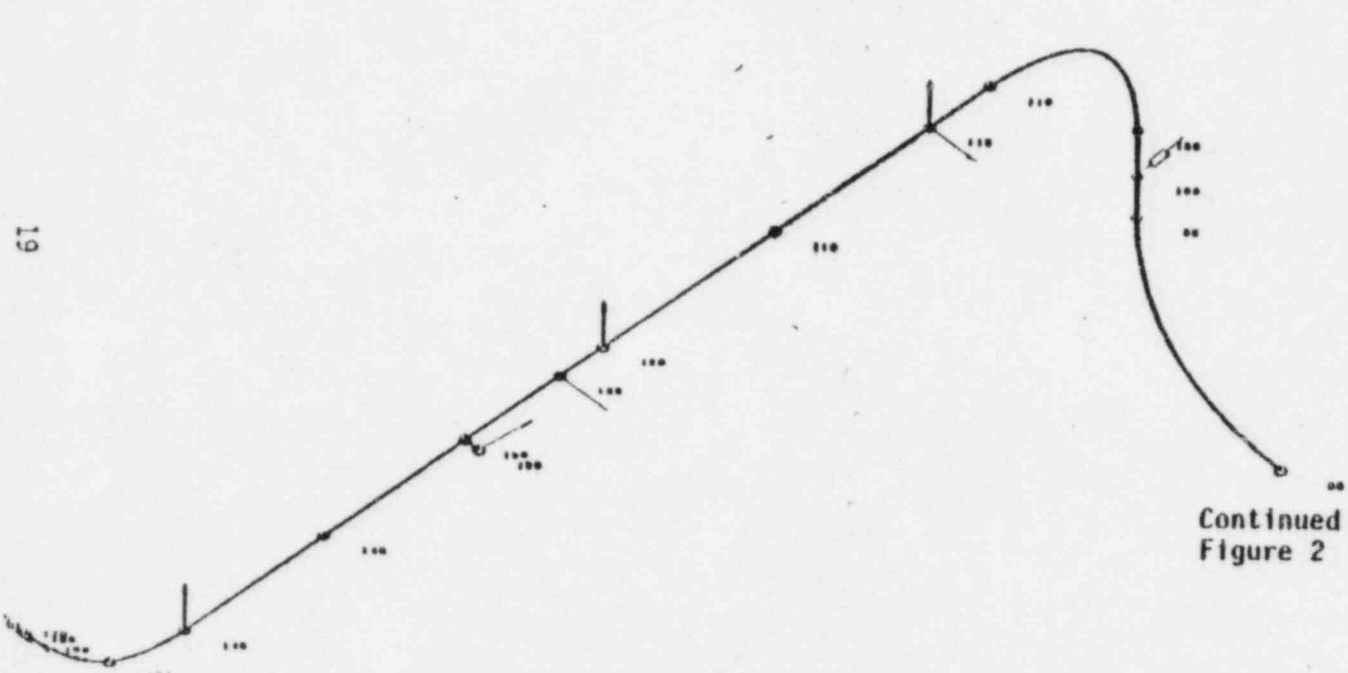


ROTATION ABOUT Y-AXIS = 48.4 DEG.
X-Z PLANE TILT = 48 DEG.

Figure 2. Partial isometric of the SI-51 computer model.

••LEGEND••

- / - NODE LOCATION
- 0 - HANGPOINT LOCATION
- W- - SPRING HANGER
- || - SNUDDER
- | - RIGID SUPPORT
- | - ANCHOR
- X - ELASTIC JOINT
- | - FLEXIBLE ANCHOR
- 0- - VALVE



ROTATION ABOUT Y-AXIS = 45.4 DEG.
 X-Z PLANE TILT = 45 DEG.

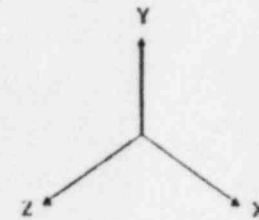
Continued on Figure 4

Continued on Figure 2

Figure 3. Partial isometric of the SI-51 computer model.

••LEGEND••

- / - NODE LOCATION
- ⊙ - NODAL POINT LOCATION
- ⌋ - SPRING HANGER
- ⌈ - SHROUD
- ⌋ - RIGID SUPPORT
- ⌈ - ANCHOR
- * - ELASTIC JOINT
- ⌈ - FLEXIBLE ANCHOR
- ⌋ - VALVE



ROTATION ABOUT Y-AXIS = 46.4 DEG.
X-Z PLANE TILT = 46 DEG.

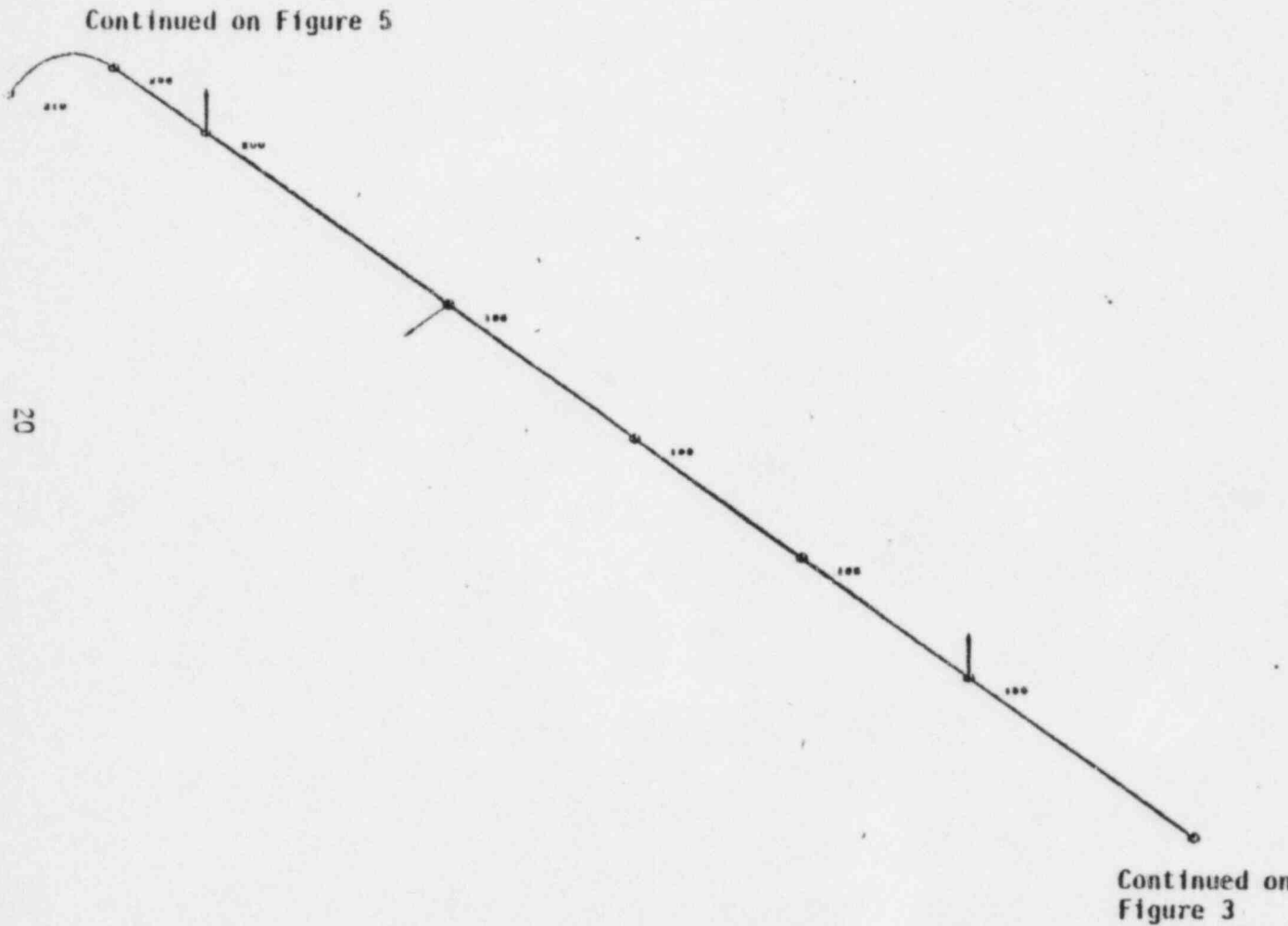
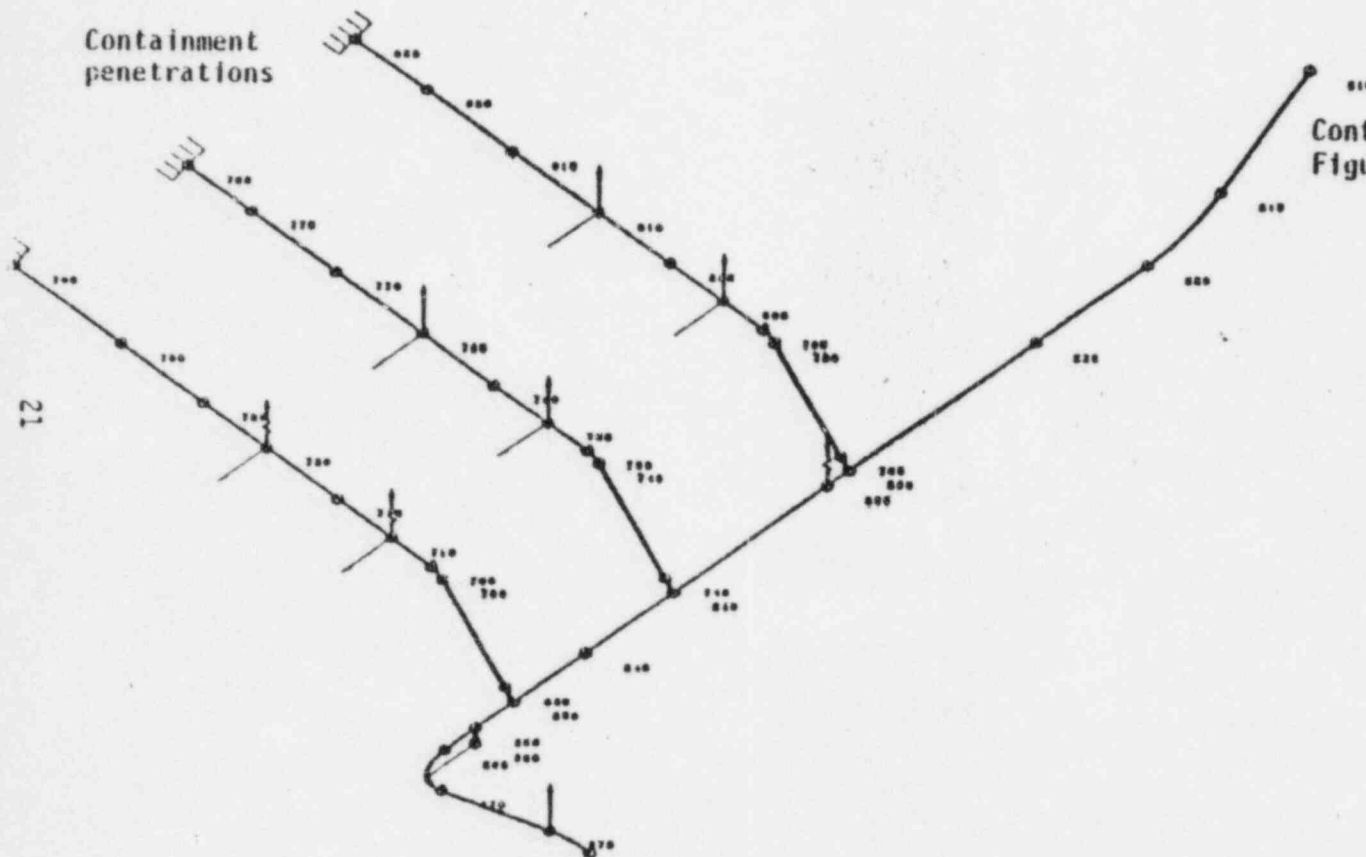


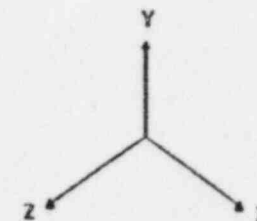
Figure 4. Partial isometric of the SI-51 computer model.

••LEGEND••

- / - NODE LOCATION
- - MASSPOINT LOCATION
- ⌋ - SPRING HANGER
- - SHOULDER
- ⌋ - RIGID SUPPORT
- ⌋ - ANCHOR
- ✱ - ELASTO JOINT
- ⌋ - FLEXIBLE ANCHOR
- ⌋ - VALVE



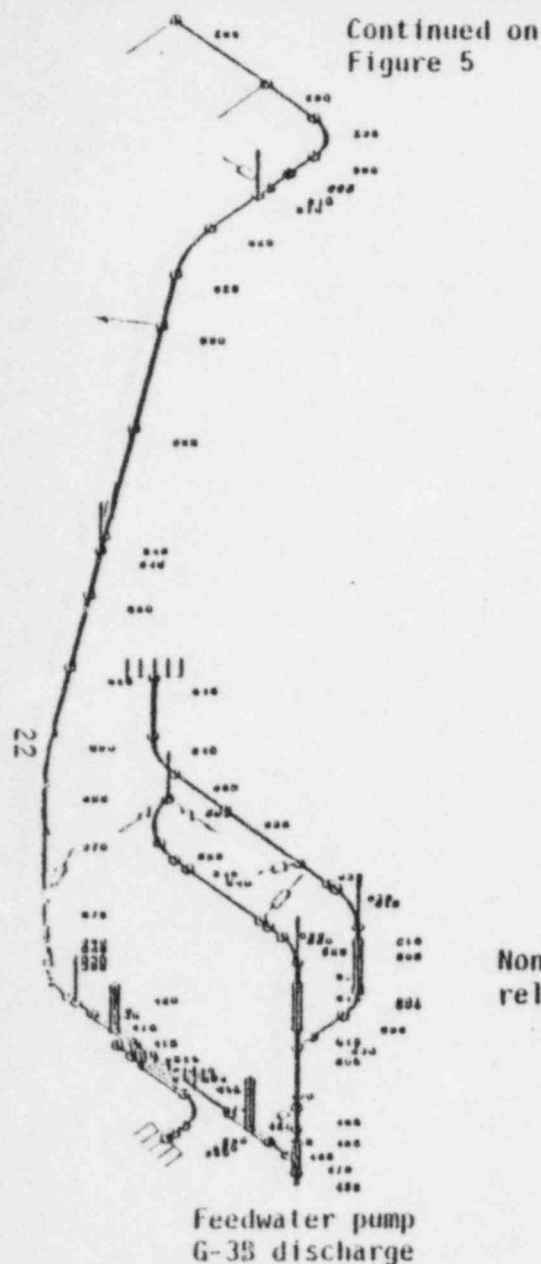
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Figure 4



ROTATION ABOUT Y-AXIS = 45.4 DEG.
X-Z PLANE TILT = 45 DEG.

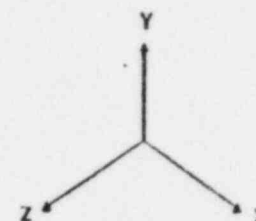
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Figure 6

Figure 5. Partial isometric of the SI-51 computer model.



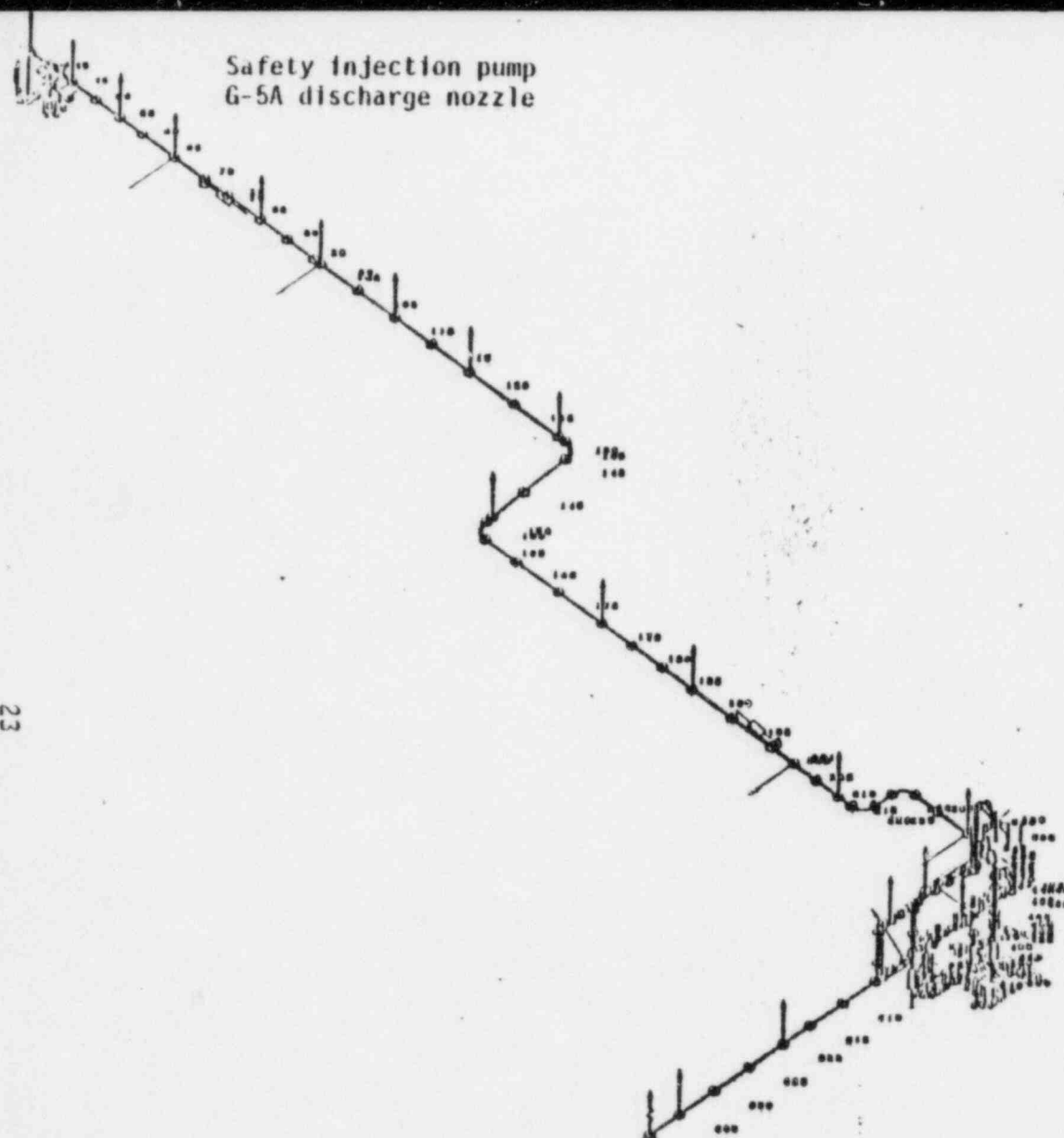
Non-safety
related

- LEGEND••
- / - NODE LOCATION
 - - MASSPOINT LOCATION
 - ~ - SPRING HANGER
 - - SNUBBER
 - ⊥ - RIGID SUPPORT
 - ⊥ - ANCHOR
 - * - ELASTIC JOINT
 - ⊥ - FLEXIBLE ANCHOR
 - ⊥ - VALVE



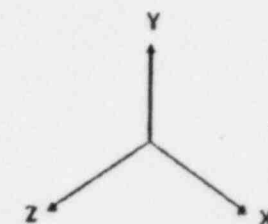
ROTATION ABOUT Y-AXIS = 45.4 DEG.
X-Z PLANE TILT = 45 DEG.

Figure 6. Partial isometric of the SI-51 computer model.



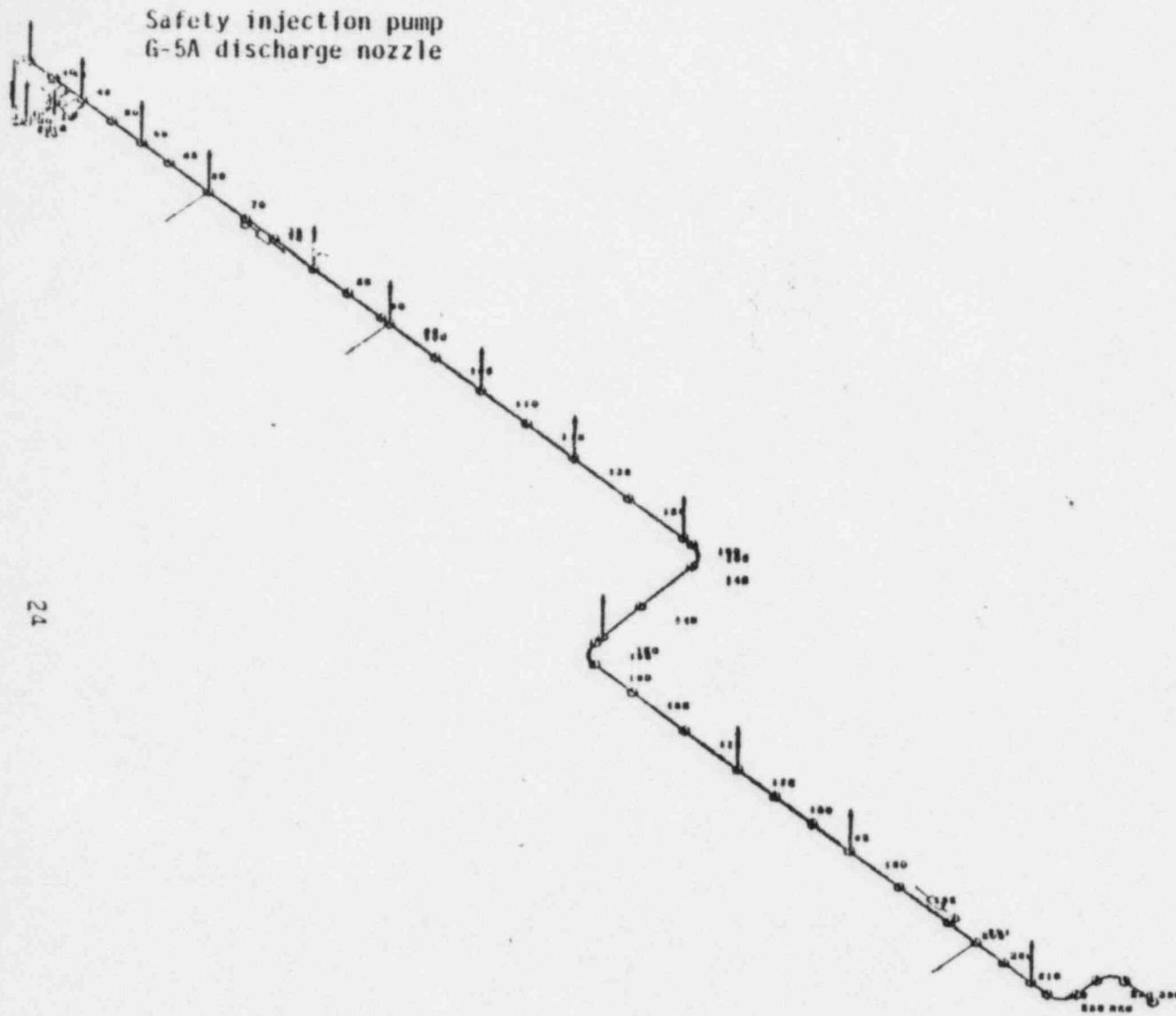
••LEGEND••

- / - NODE LOCATION
- - MASSPOINT LOCATION
- ⚡ - SPRING HANGER
- - SNUBBER
- ⌋ - RIGID SUPPORT
- |— - ANCHOR
- * - ELASTIC JOINT
- |— - FLEXIBLE ANCHOR
- ⌋ - VALVE

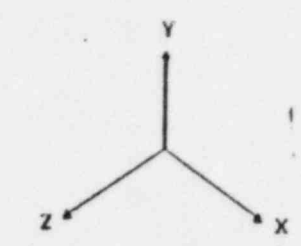


ROTATION ABOUT Y-AXIS = 45.4 DEG.
X-Z PLANE TILT = 45 DEG.

Figure 7. Isometric of the SI-04 computer model.



- LEGEND••
- / - NODE LOCATION
 - - MASSPOINT LOCATION
 - ~ - SPRING HANGER
 - - SHOCKER
 - ⌋ - RIGID SUPPORT
 - ⌋ - ANCHOR
 - * - ELASTIC JOINT
 - ⌋ - FLEXIBLE ANCHOR
 - ⌋ - VALVE



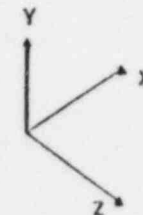
ROTATION ABOUT Y-AXIS = 40.4 DEG.
X-Z PLANE TILT = 46 DEG.

Continued on
Figure 9

Figure 8. Partial isometric of the SI-04 computer model.

••LEGEND••

- / - NODE LOCATION
- ⊙ - NADPOINT LOCATION
- ~ - SPRING HANGER
- [] - SMOOTHER
- |— - RIGID SUPPORT
- |— - ANCHOR
- * - ELASTIC JOINT
- |— - FLEXIBLE ANCHOR
- |— - VALVE



ROTATION ABOUT Y-AXIS = 44.0 DEG.
X-Z PLANE TILT = 46 DEG.

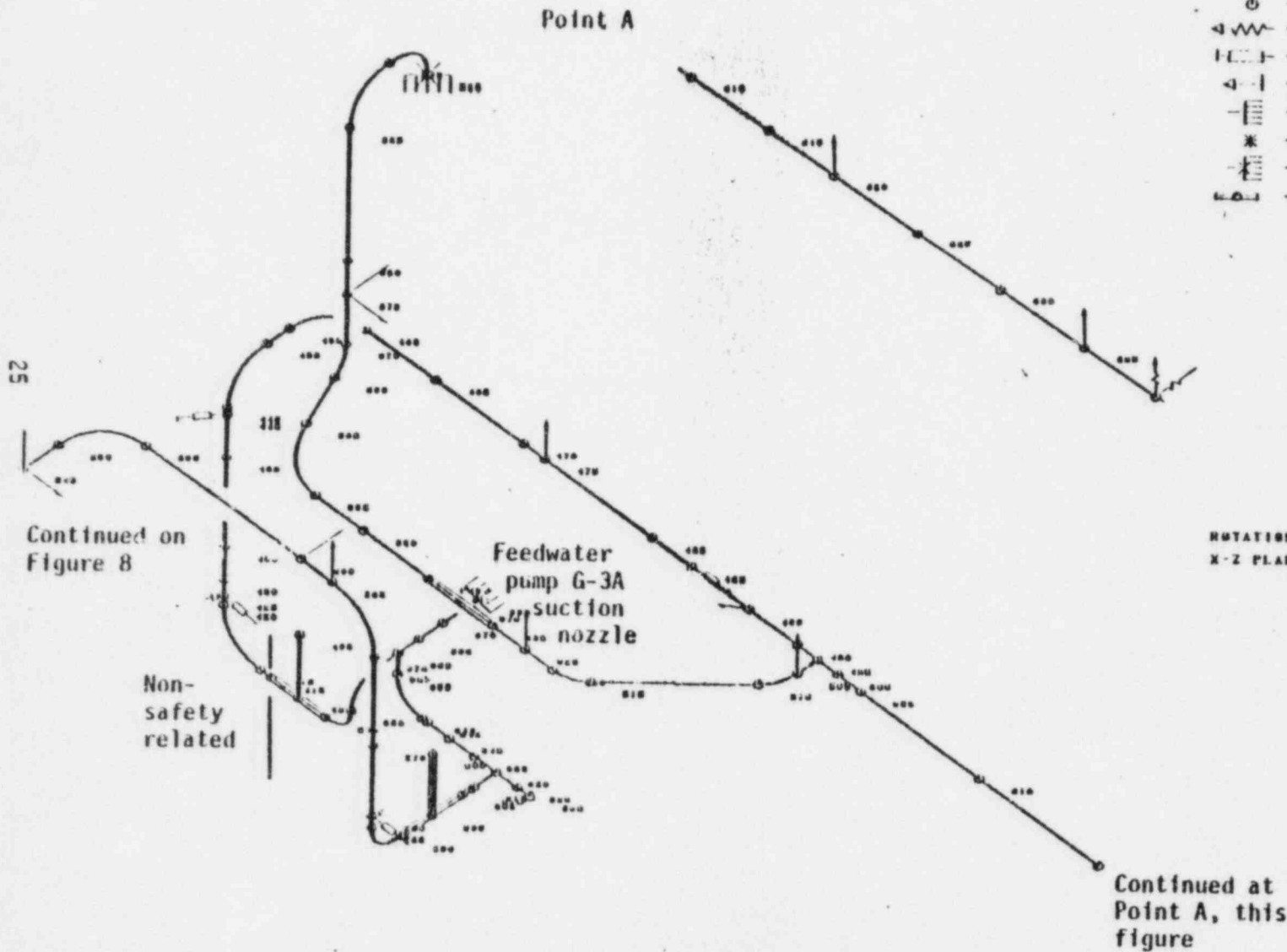
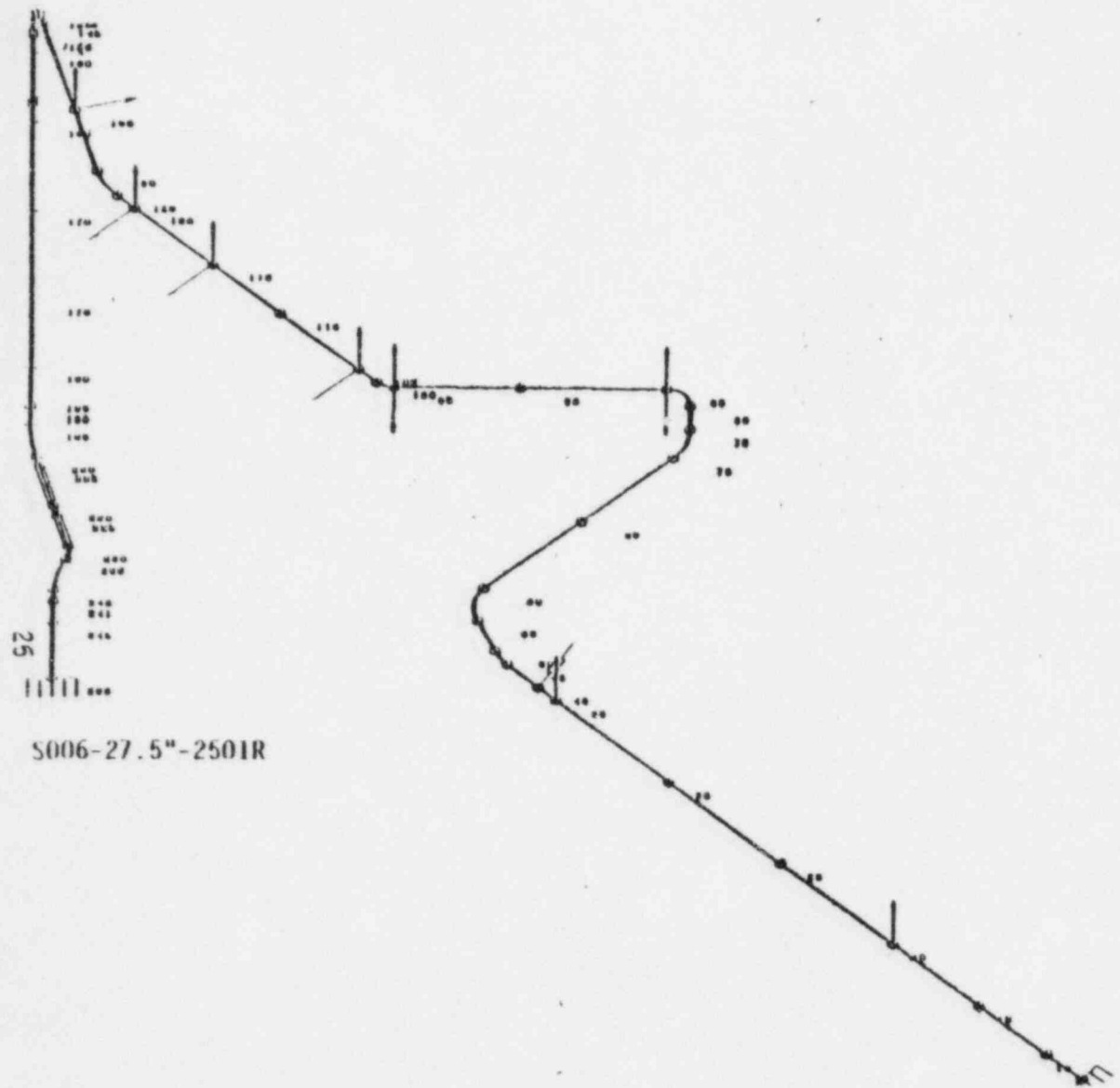
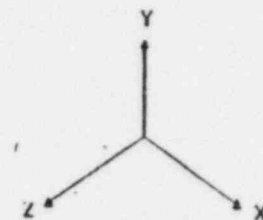


Figure 9. Partial isometric of the SI-04 computer model.



••LEGEND••

- / - NODE LOCATION
- ⊙ - MASSPOINT LOCATION
- ⚡ - SPRING HANGER
- ⌈⌋ - SHOCKER
- ⌈⌋ - RIGID SUPPORT
- ⌈⌋ - ANCHOR
- * - ELASTIC JOINT
- ⌈⌋ - FLEXIBLE ANCHOR
- ⌈⌋ - VALVE



ROTATION ABOUT Y-AXIS = 46.4 DEG.
X-Z PLANE TILT = 46 DEG.

S006-27.5"-2501R

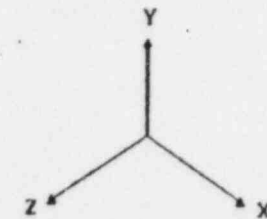
Containment penetration
(continued on Figure 5)

Figure 10. Isometric of the SI-158 computer model.

Continued on Figure 12

••LEGEND••

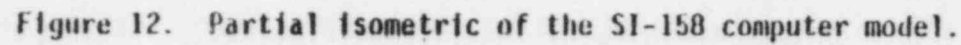
- / - NODE LOCATION
- ⊙ - MASSPOINT LOCATION
- ⌋ - SPRING HANGER
- ⌊ - SHOCKER
- ⌋ - RIGID SUPPORT
- ⌊ - ANCHOR
- * - ELASTIC JOINT
- ⌊ - FLEXIBLE ANCHOR
- ⌊ - VALVE



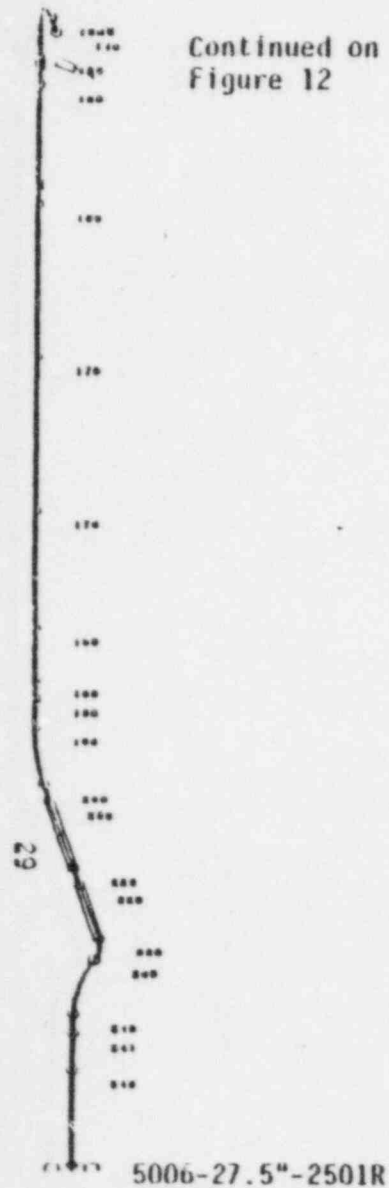
ROTATION ABOUT Y-AXIS = 45.4 DEG.
X-Z PLANE TILT = 45 DEG.

Containment
penetration

Figure 11. Partial isometric of the SI-158 computer model.

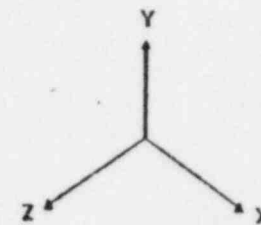


Continued on
Figure 12



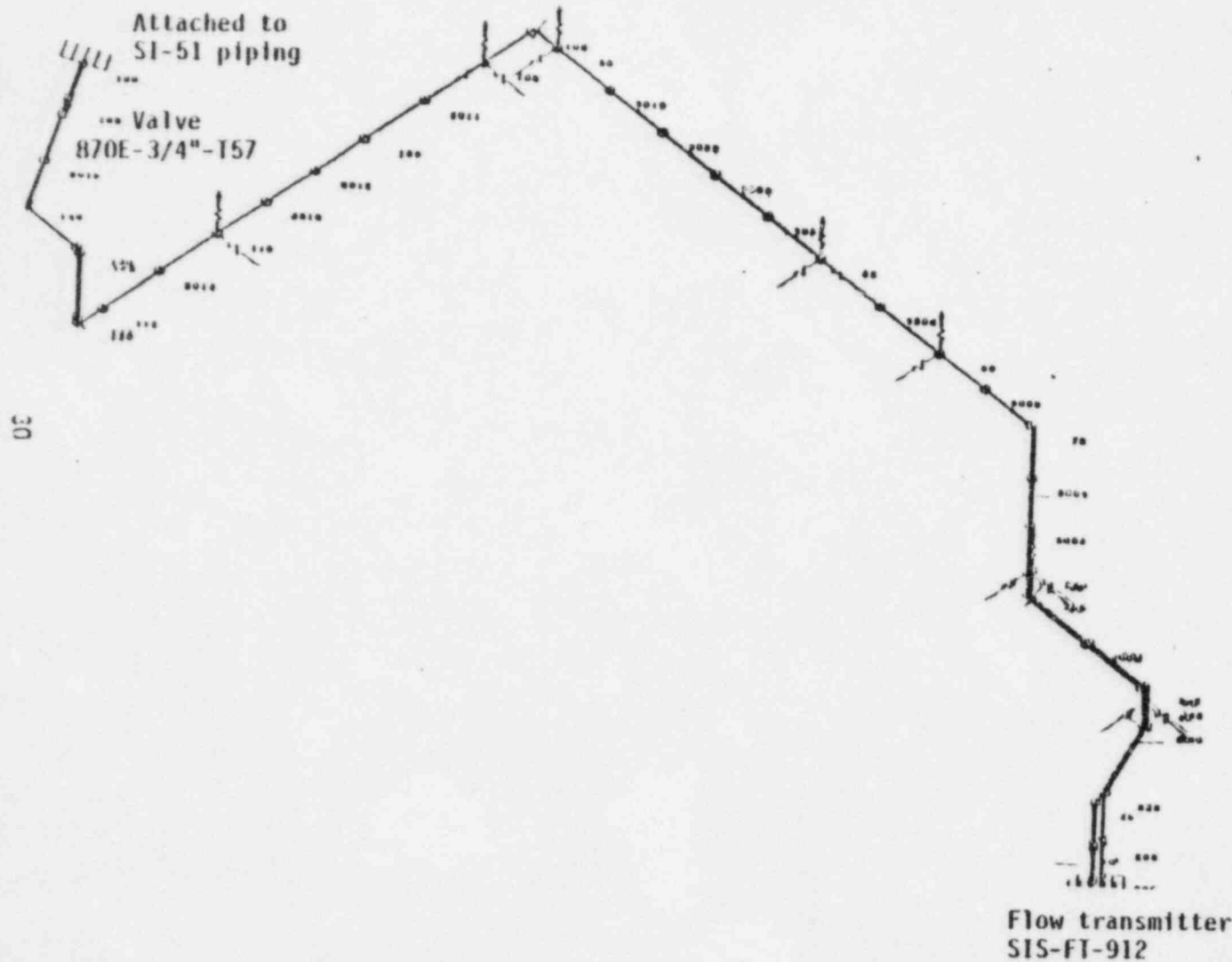
••LEGEND••

- / - NODE LOCATION
- ⊙ - MASSPOINT LOCATION
- ~ - SPRING HANGER
- - SHOCKER
- ⌋ - RIGID SUPPORT
- ≡ - ANCHOR
- ✱ - ELASTO JOINT
- ≡ - FLEXIBLE ANCHOR
- Valve - VALVE

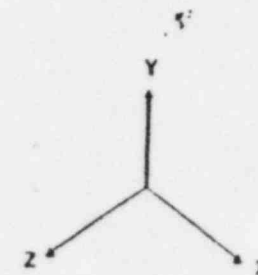


ROTATION ABOUT Y-AXIS * 45.4 DEG.
X-Z PLANE TILT * 45 DEG.

Figure 13. Partial isometric of the SI-158 computer model.



- LEGEND••
- / - NODE LOCATION
 - - MASSPOINT LOCATION
 - ~ - SPRING HANGER
 - - SNUDDER
 - |— - RIGID SUPPORT
 - E— - ANCHOR
 - X— - ELASTO JOINT
 - E— - FLEXIBLE ANCHOR
 - V— - VALVE

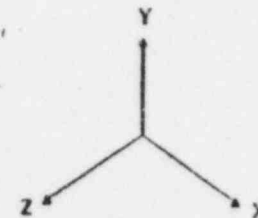


ROTATION ABOUT Y-AXIS + 45.4 DEG.
 X-Z PLANE TILT + 45 DEG.

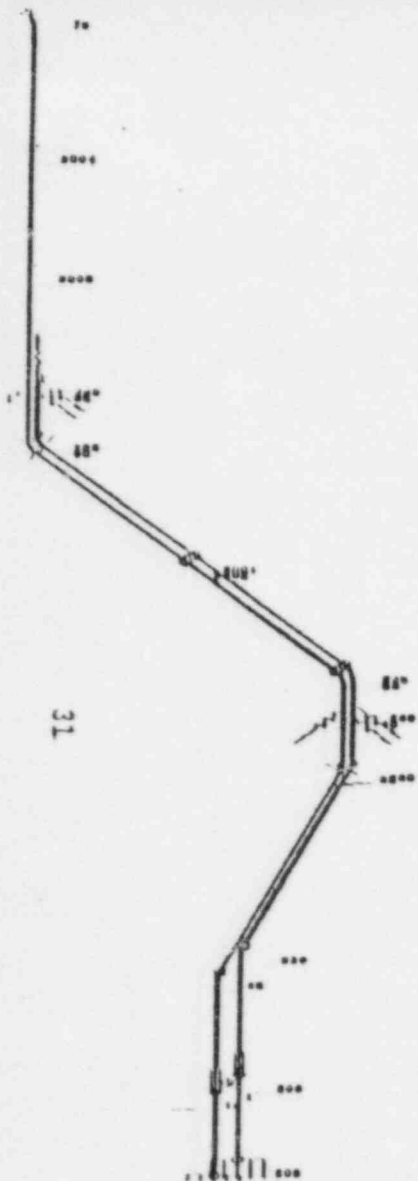
Figure 14. Isometric of the SI-51 tubing computer model.
 00/01/07.

••LEGEND••

- / - NODE LOCATION
- - NAGUPINT LOCATION
- ⌋ - SPRING HANGER
- - SHOCKER
- ⌋ - RIGID SUPPORT
- ≡ - ANCHOR
- * - ELASTIC JOINT
- ≡ - FLEXIBLE ANCHOR
- VALVE - VALVE



ROTATION ABOUT Y-AXIS = 45.4 DEG.
 X-Z PLANE TILT = 45 DEG.



Flow transmitter
 SIS-FI-912

Figure 15. Partial isometric of the SI-51 tubing computer model.

1 1YPI

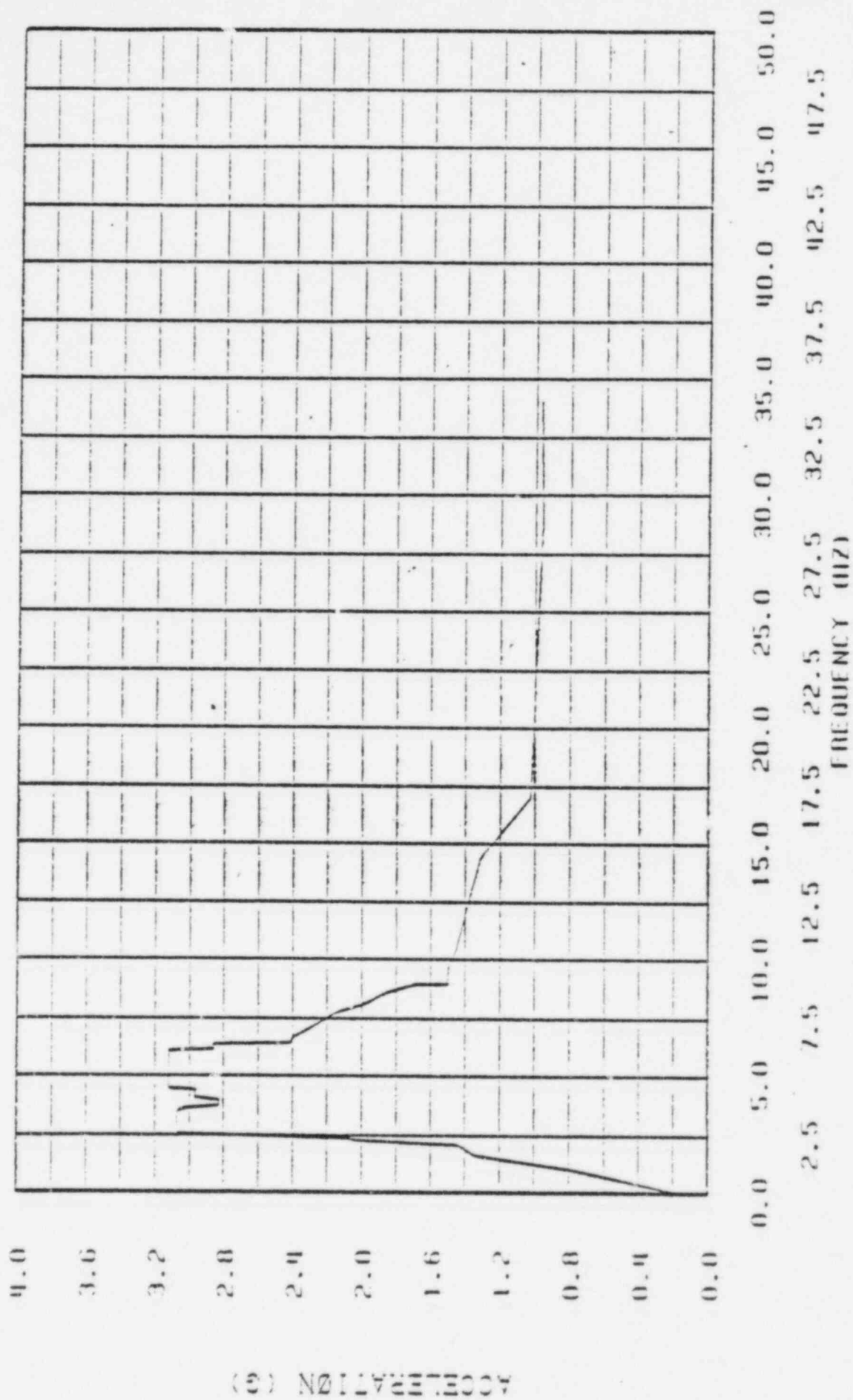


Figure 16. North-South response spectrum for SI-51.

1 TYP1

33

ACCELERATION (G)

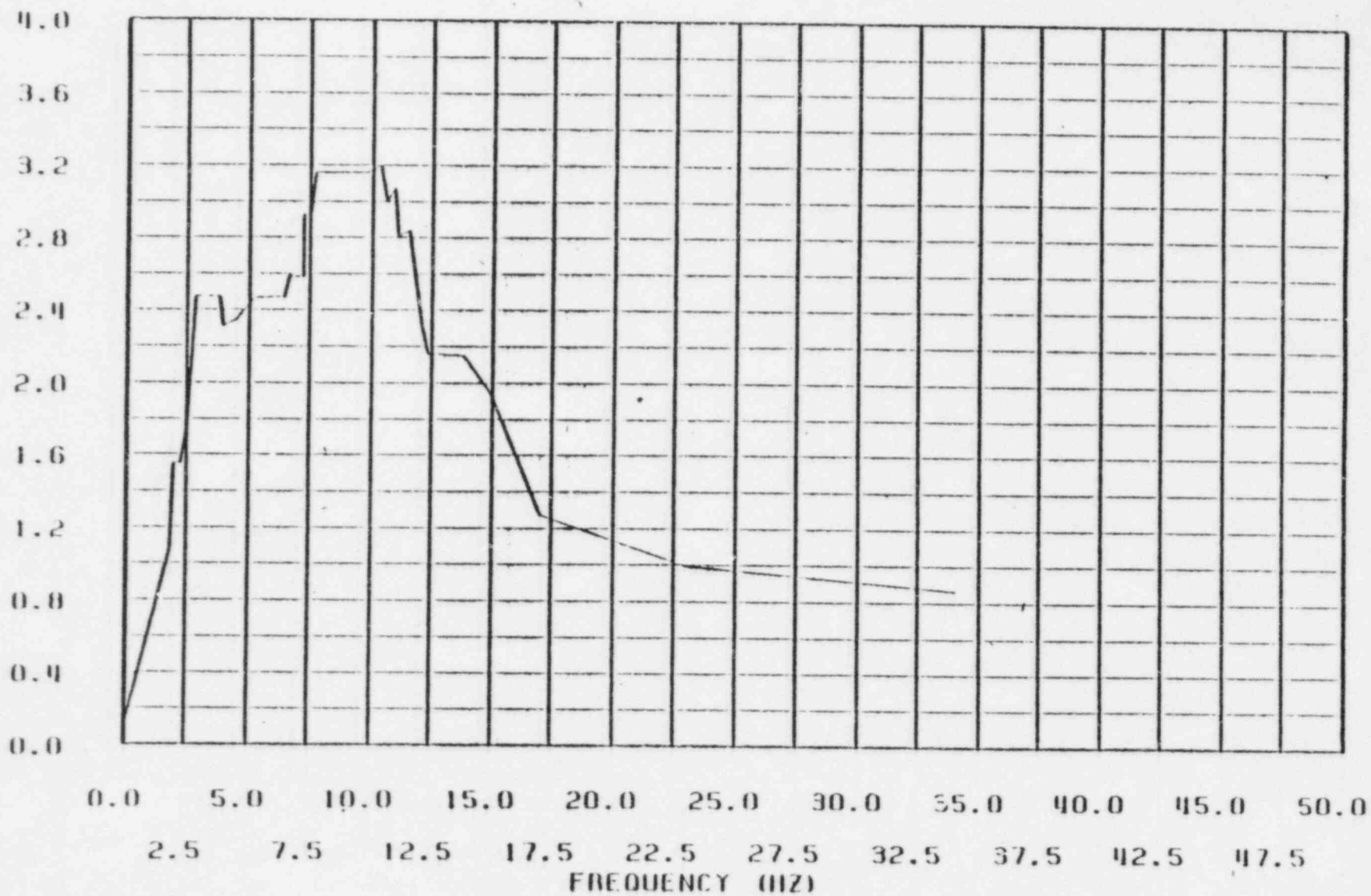


Figure 17. Vertical response spectrum for SI-51.

1 TYP

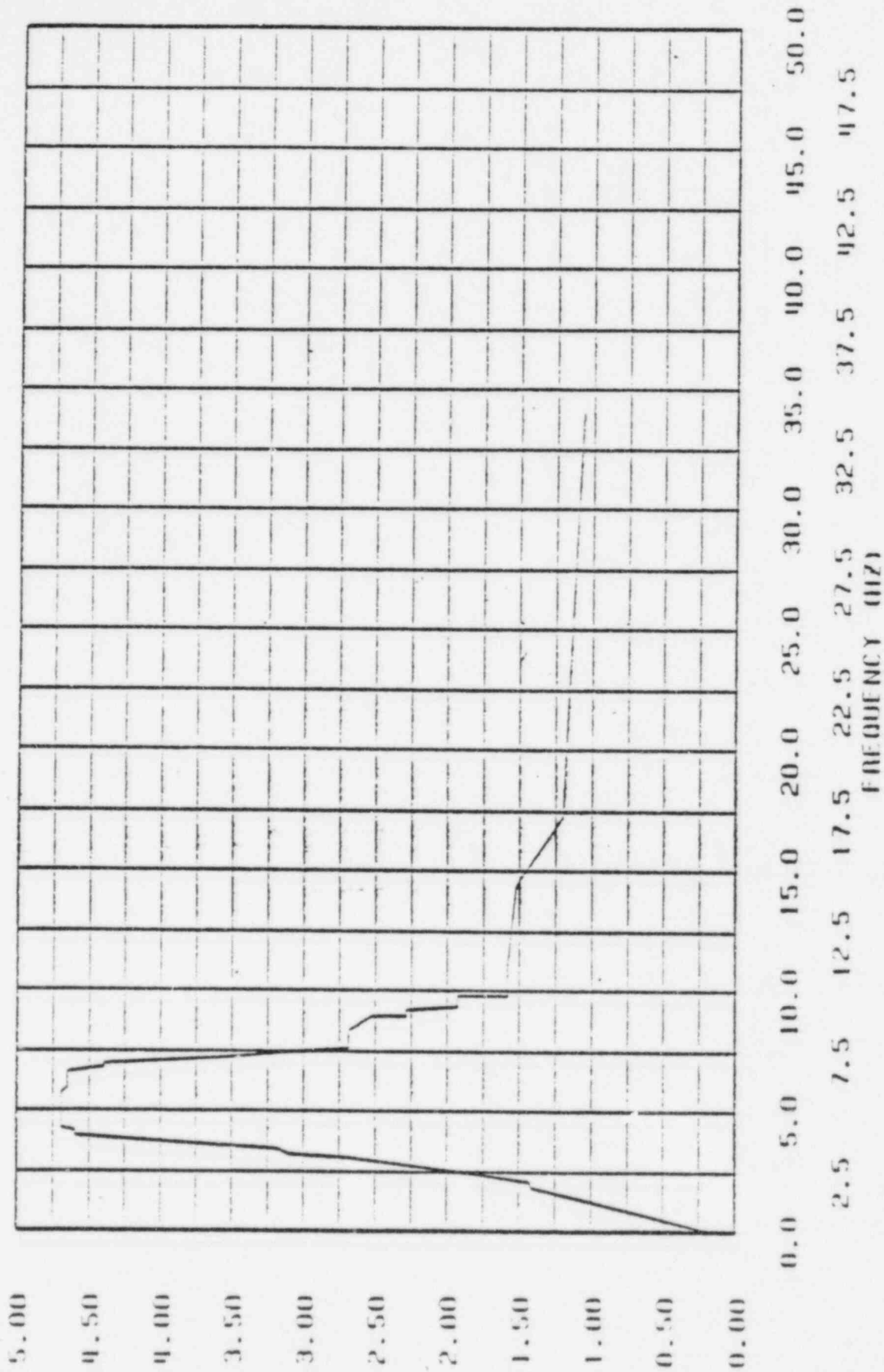


Figure 18. East-West response spectrum for SI-51.

(6) NO. 114877-1000

1 TYPE

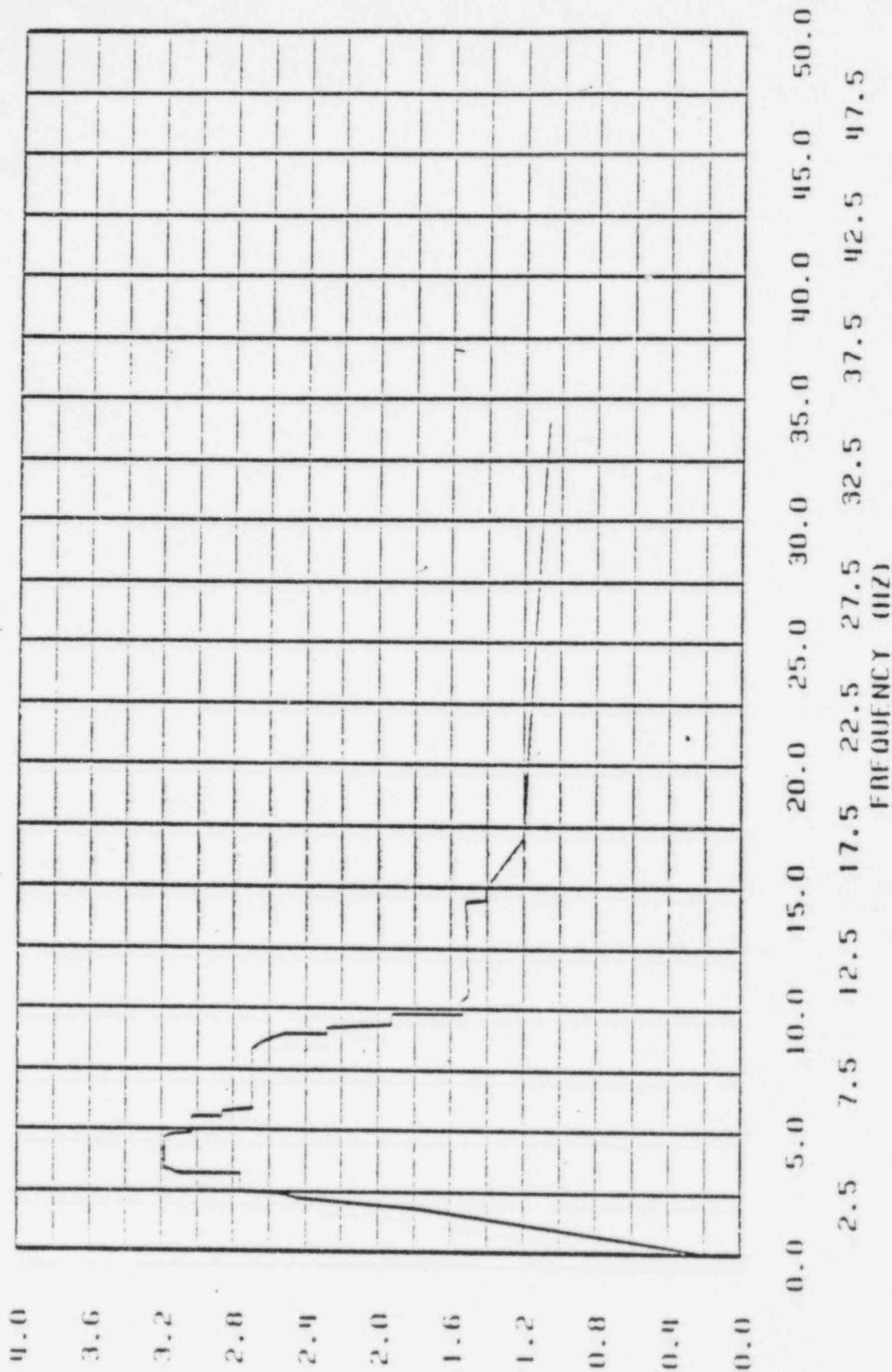


Figure 19. North-South response spectrum for SI-04.

1 TYPE

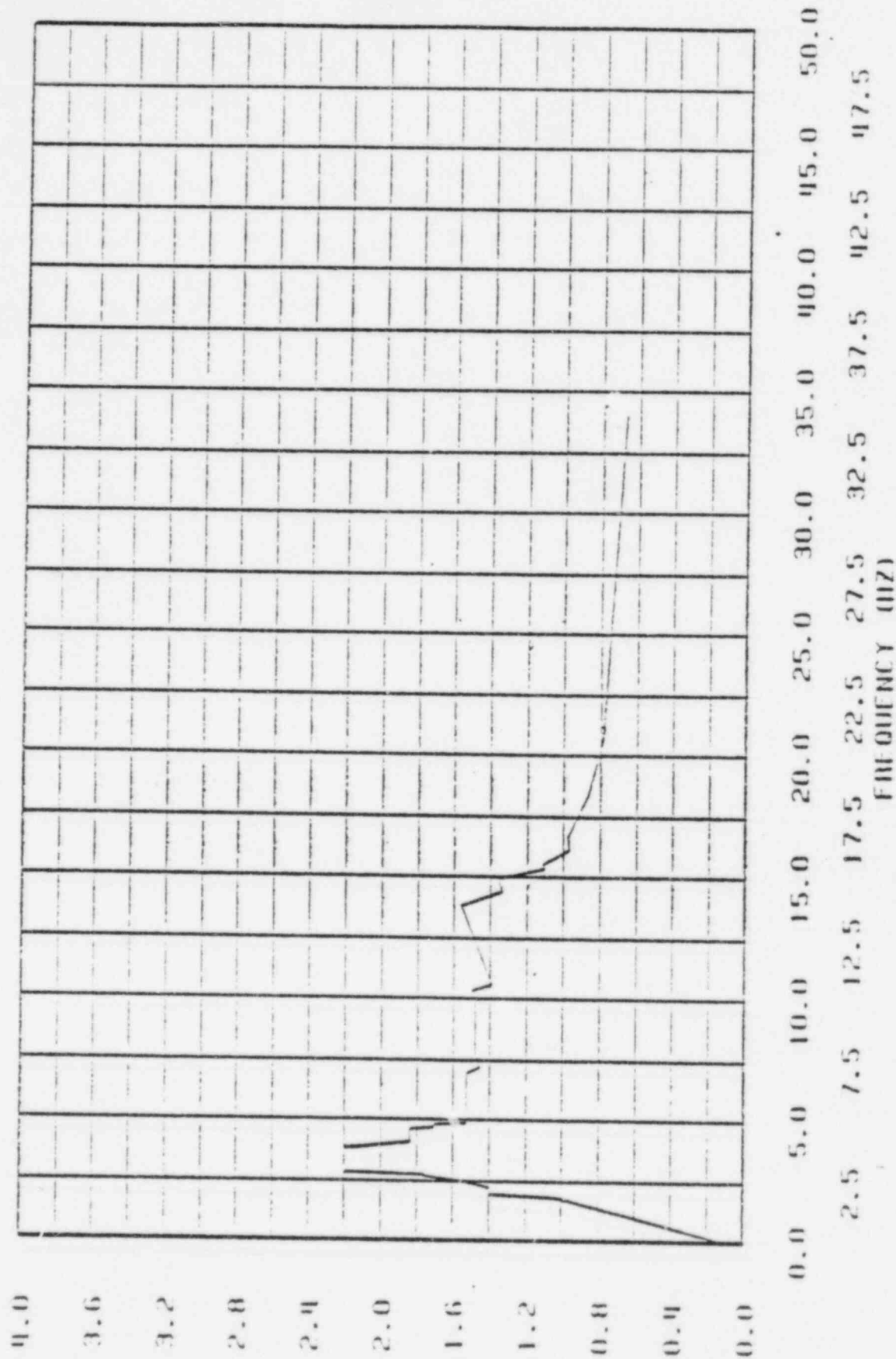


Figure 20. Vertical response spectrum for SI-04.

1 TYPE

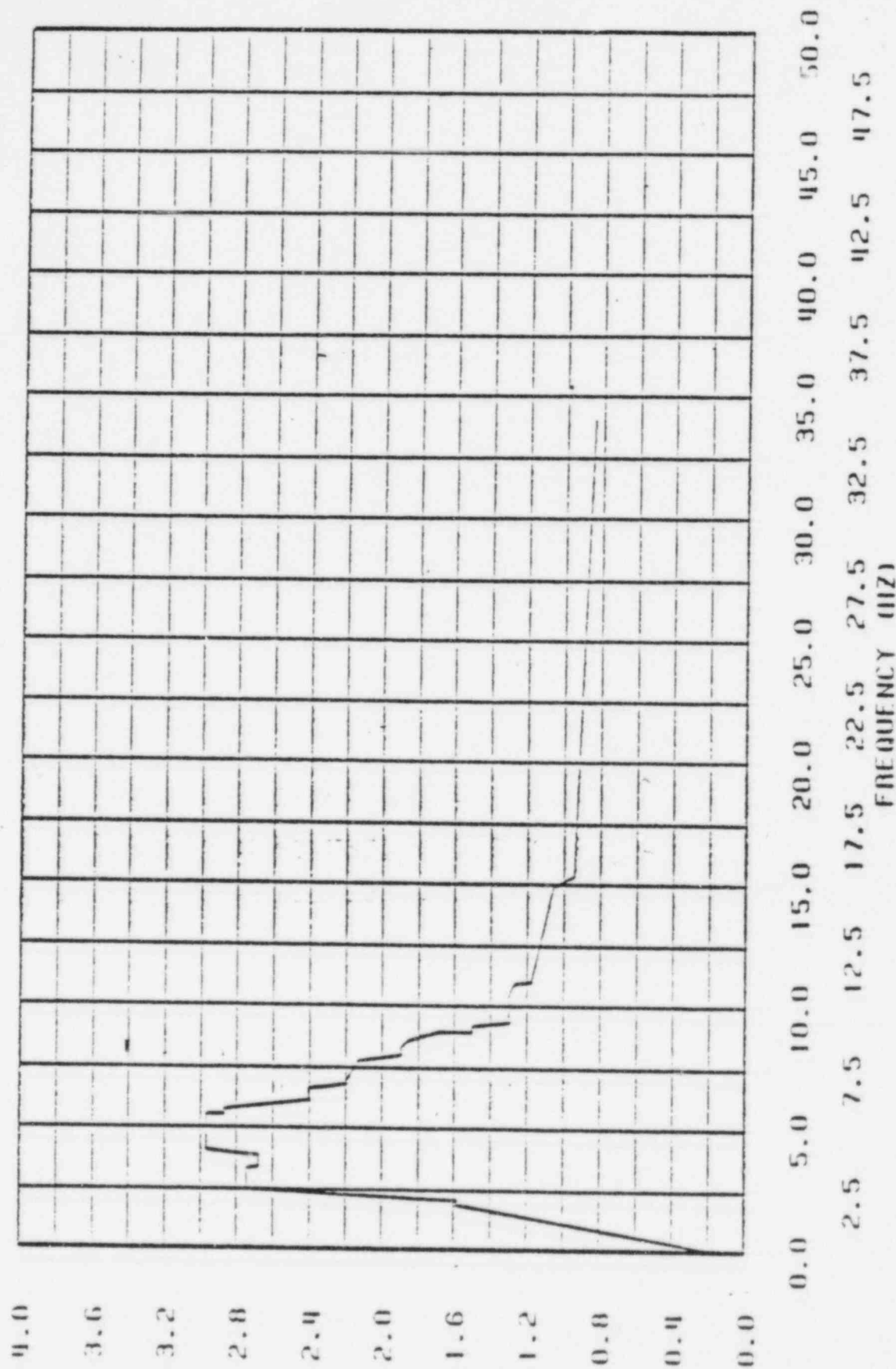


Figure 21. East-West response spectrum for S1-04.

1 TYPE

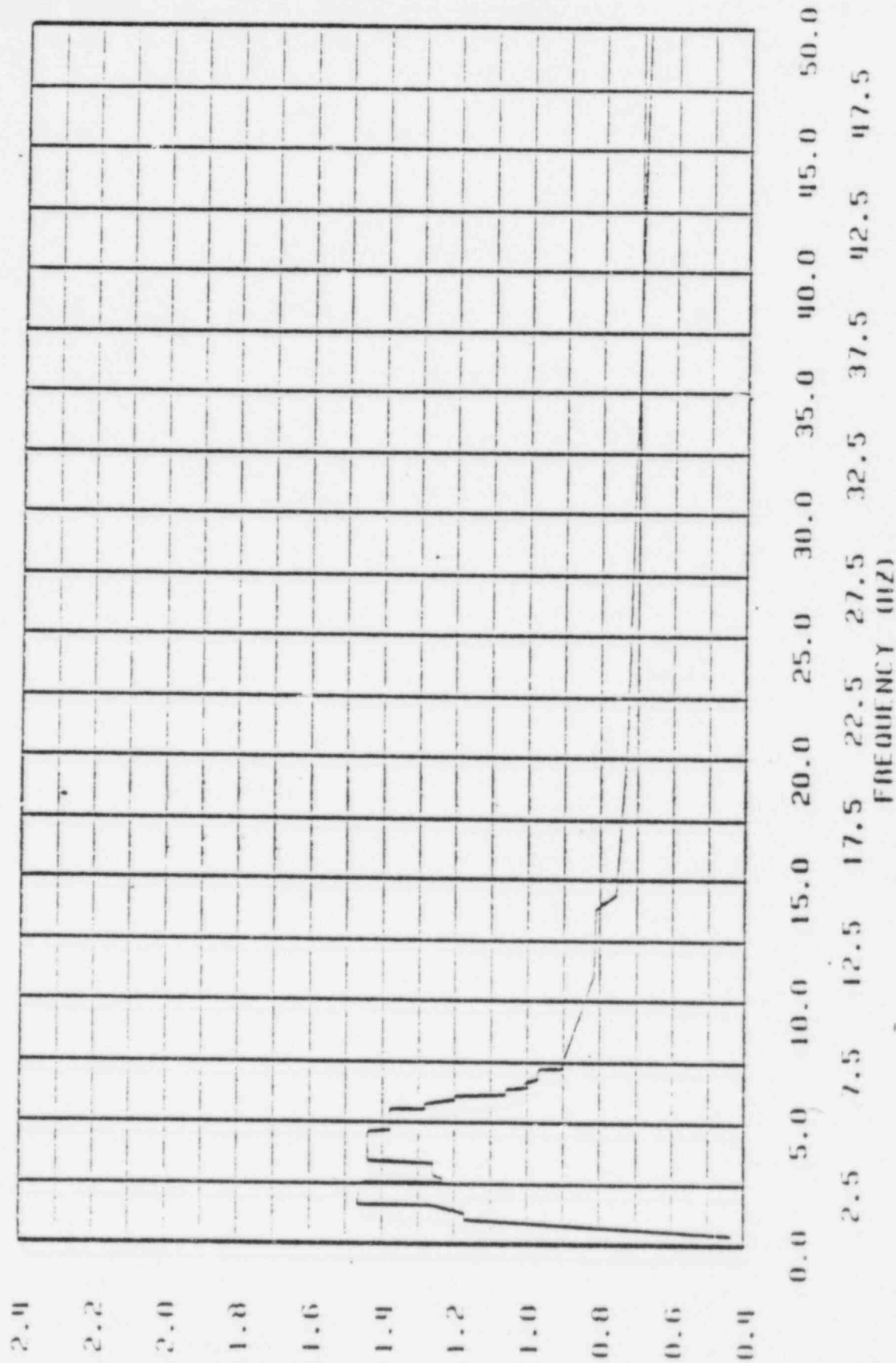


Figure 22. North-South response spectrum for SI-158.

1 TYPE

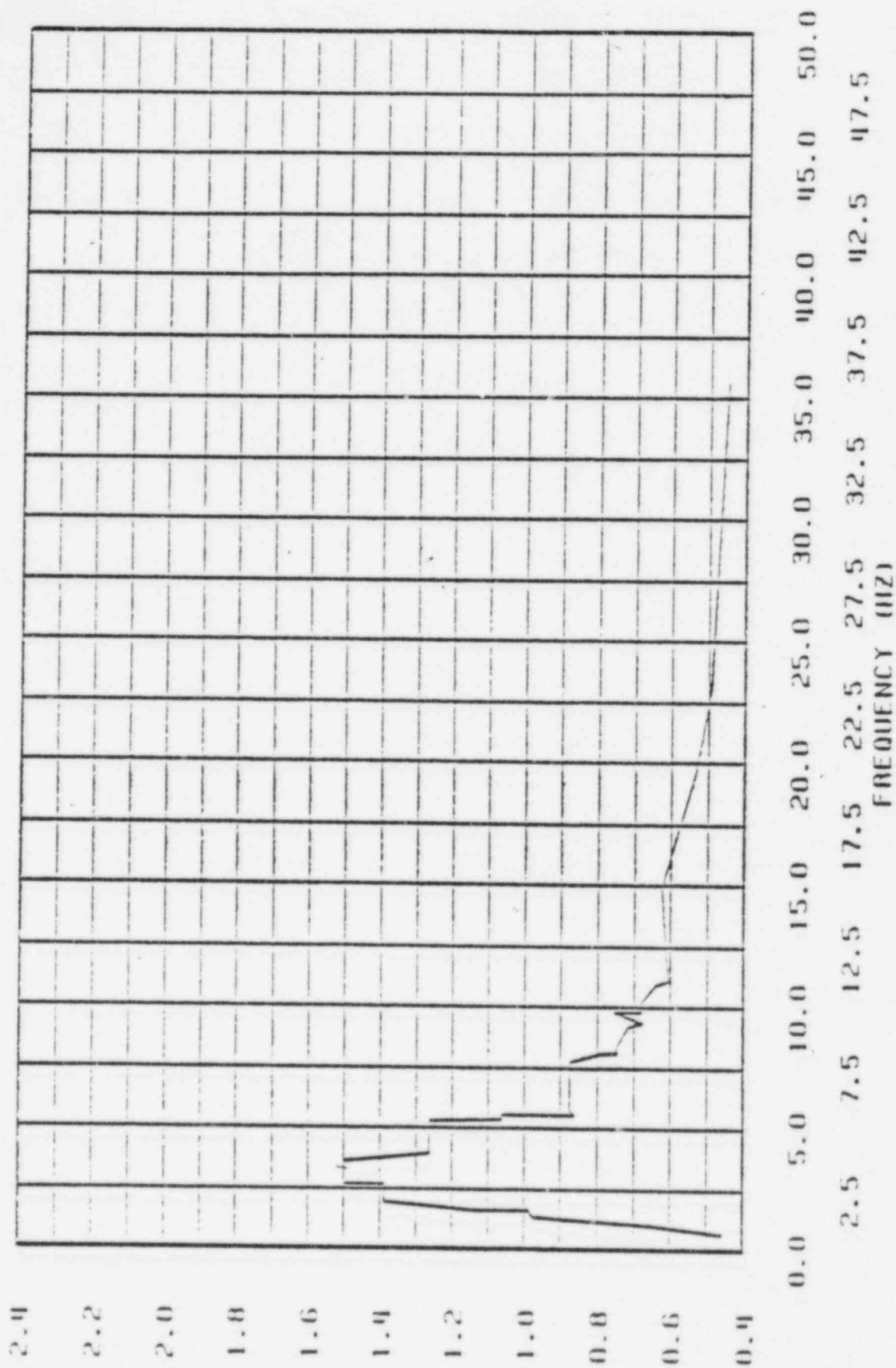


Figure 23. Vertical response spectrum for SI-158.

1 TYP

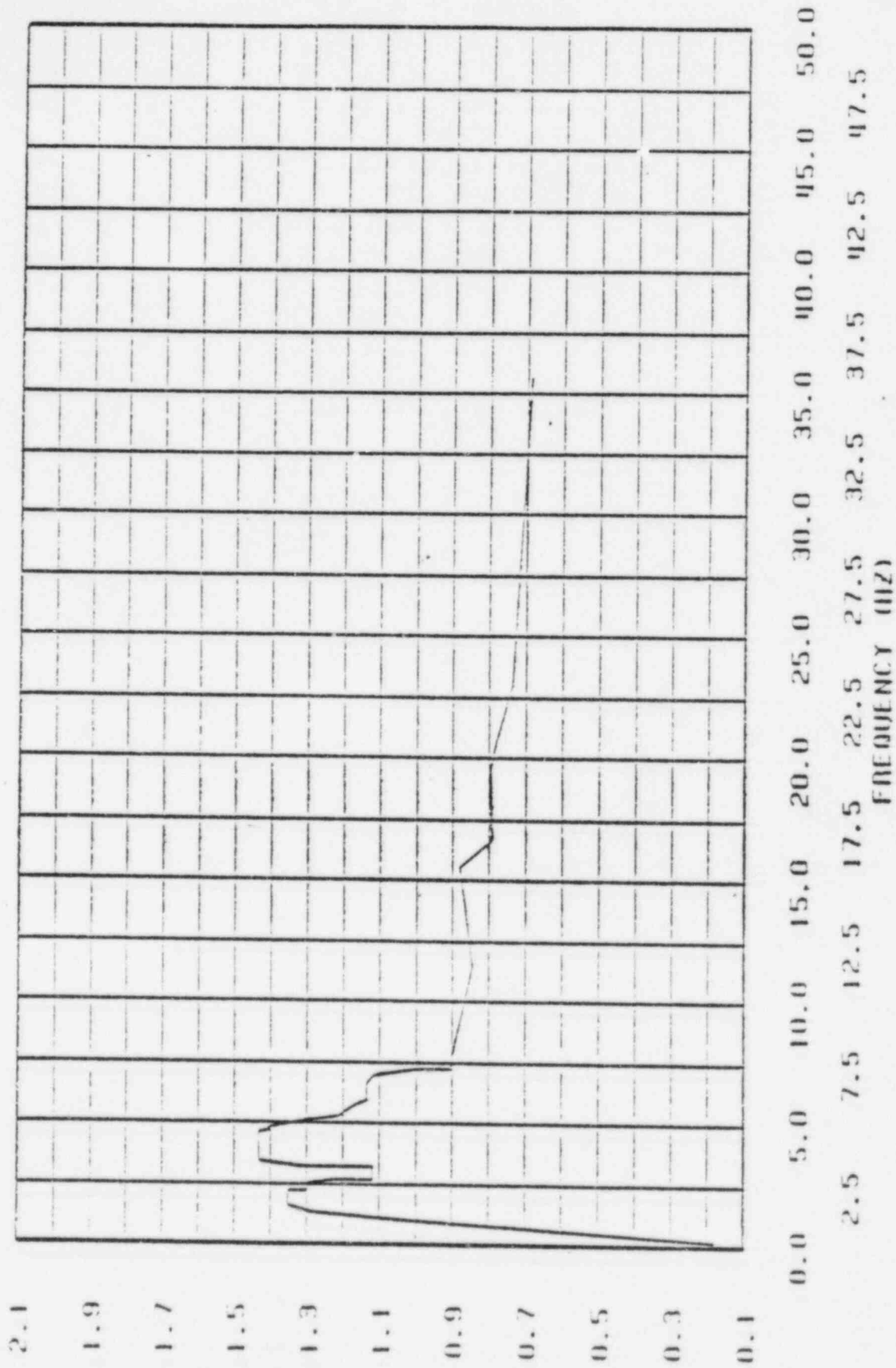


Figure 24. East-West response spectrum for SI-158.

ACCELERATION (G)

1 TYP1

41
ACCELERATION (G)

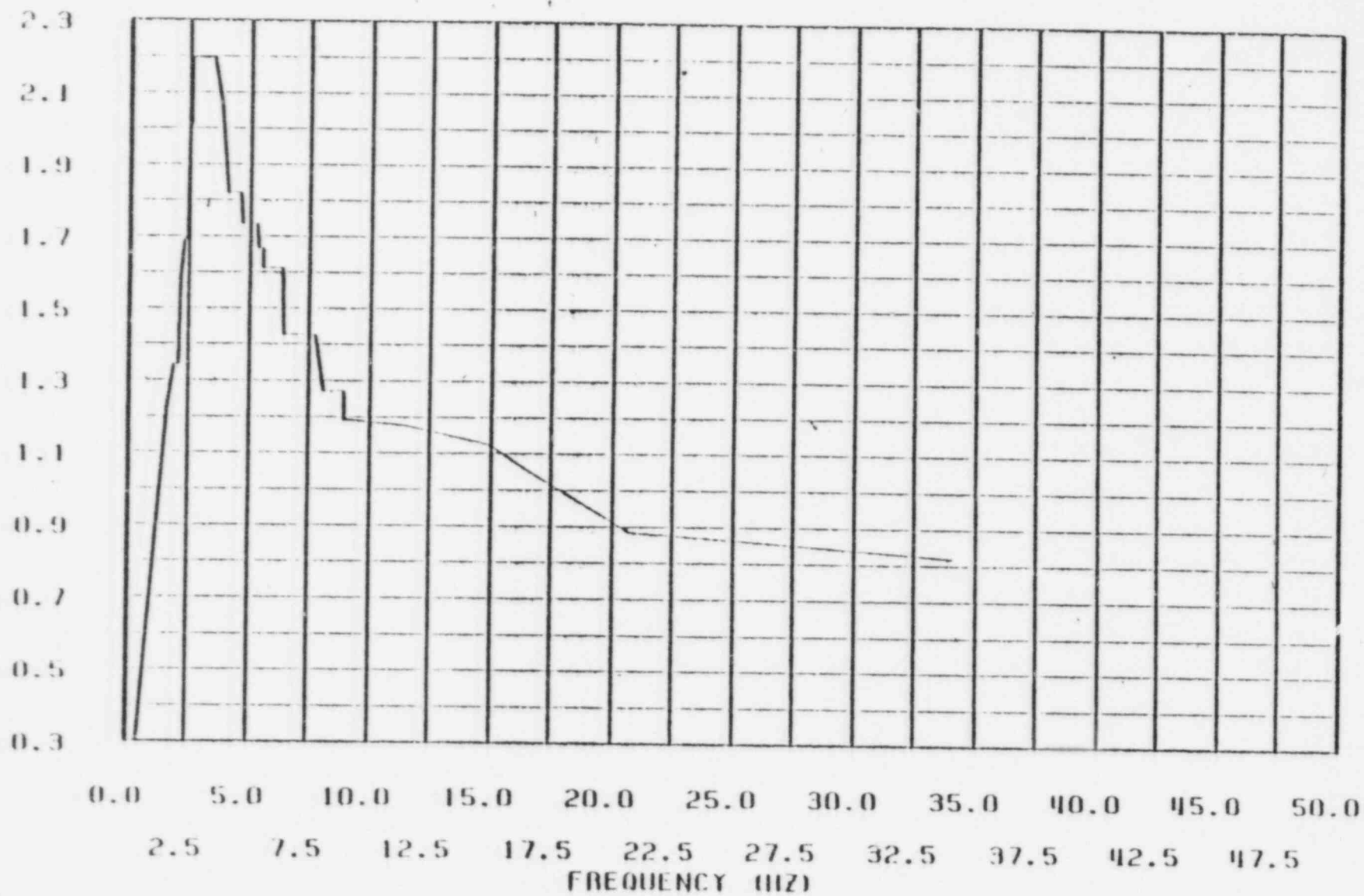


Figure 25. North-South response spectrum for SI-51 tubing.

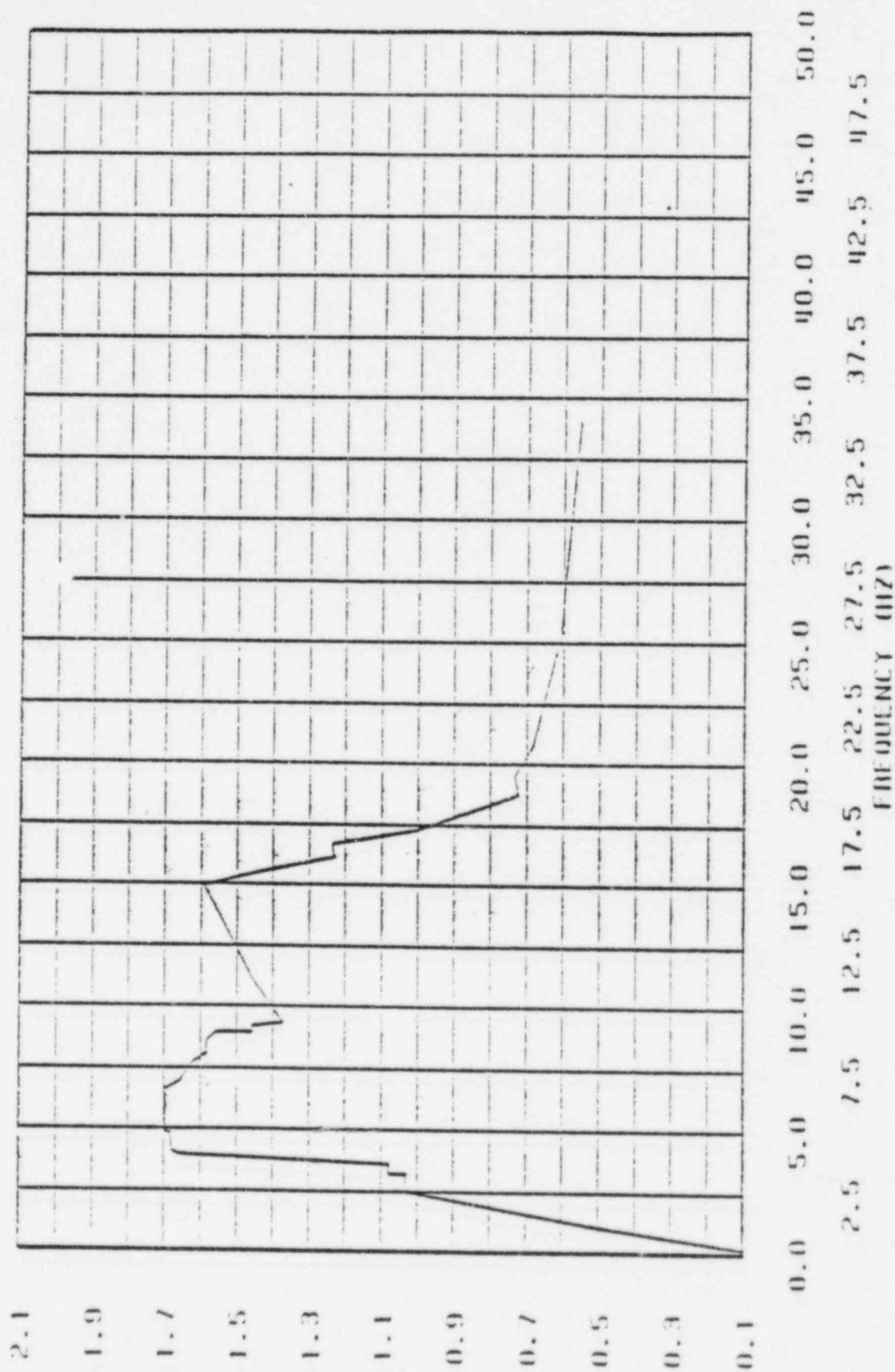


Figure 26. Vertical response spectrum for SI-51 tubing.

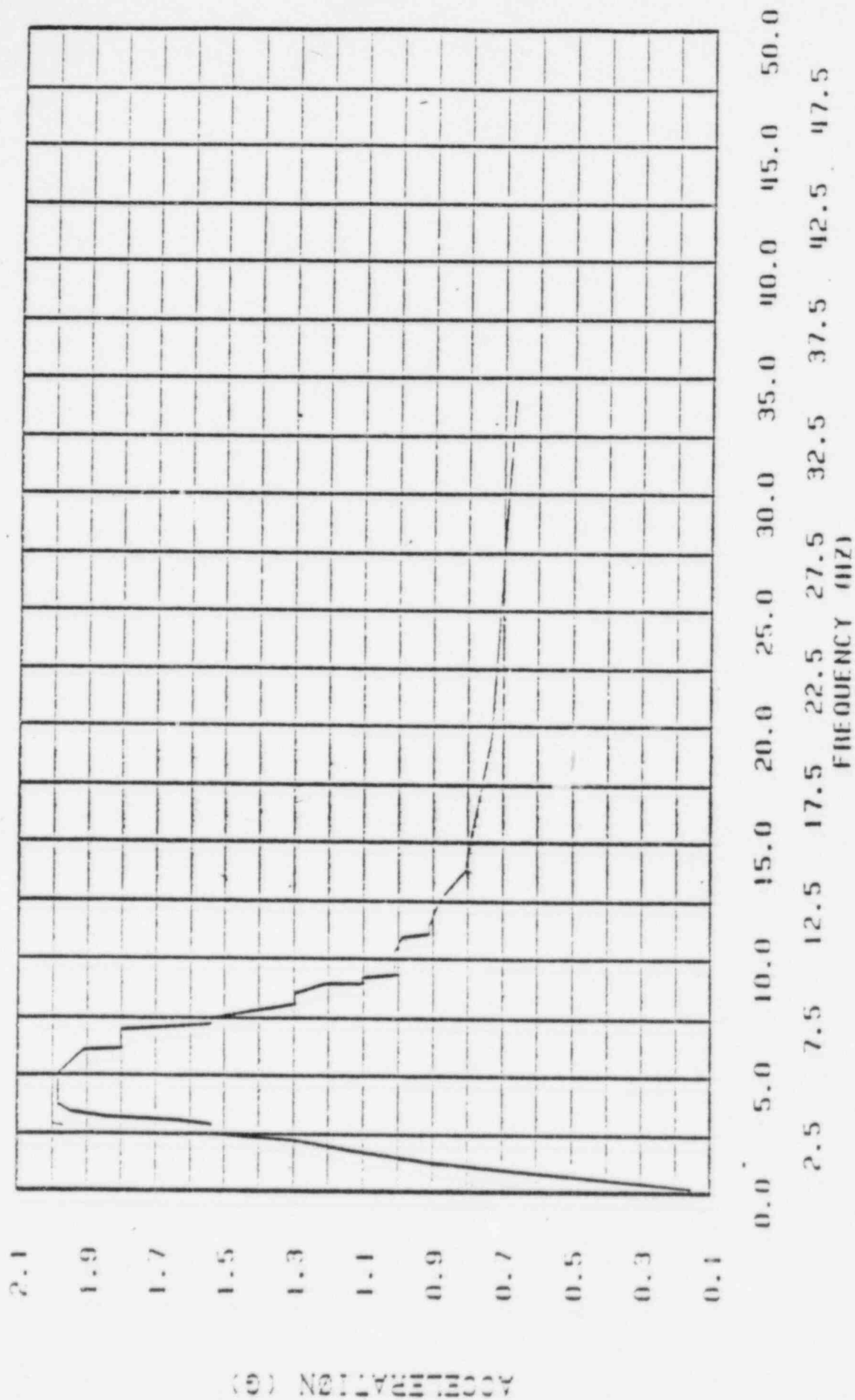


Figure 27. East-West response spectrum for SI-51 tubing.

TABLE 1. PIPE DATA AND MATERIAL SPECIFICATIONS--SIS PROBLEM SI-51

Line number	6006-6-1501R 6007-6-1501R 6008-6-1501R	319-12-EG 320-12-EG	6004-14-1501R 6005-14-1501R	319-10-EG 320-10-EG
OD (inches)	6.625	12.75	14.0	10.75
Thickness (inches)	0.432	0.687	0.58	0.593
Weight ^a (lb/ft)	39.86	140.12	139.27	101.97
Material	A312, TP-316	A106-B	A312, TP-316	A106-B
E _{cold} (10 ⁶ psi)	28.3	27.9	28.3	27.9
S _y (psi)	18800	15000	18800	15000
Design temperature (°F)	120	340	120	340
Operating temperature (°F)	120	340	120	340
Design pressure (psi)	900	1350	900	1350
Operating pressure (psi)	900	1350	900	1350

a. Includes weight of pipe, contents, and insulation.

TABLE 2. PIPE DATA AND MATERIAL SPECIFICATIONS--SIS PROBLEM SI-04

Line number	6002-16-151R	318-14-GG	338-8-GG
OD (inches)	16.0	14.0	8.625
Thickness (inches)	0.375	0.375	0.322
Weight ^a (lb/ft)	141.68	122.38	50.24
Material	A312, TP-304	A53-B	A53-B
E _{cold} (10 ⁶ psi)	28.3	27.9	27.9
S _h (psi)	18600	15000	15000
Design temperature (°F)	120	340	95
Operating temperature (°F)	120	340	95
Design pressure (psi)	140	360	360
Operating pressure (psi)	140	360	360

a. Includes weight of pipe, contents, and insulation.

TABLE 3. PIPE DATA AND MATERIAL SPECIFICATIONS--SIS PROBLEM SI-138

Line number	6007-6-1501R	6007-6-2501R
OD (inches)	6.625	6.625
Thickness (inches)	0.432	0.718
Weight ^a (lb/ft)	39.86	60.46
Material	SA312, TP-316	SA312, TP-316
E_{col_d} (10^6 psi)	28.3	28.3
S_h (psi)	18800	18800
Design temperature (°F)	120	570
Operating temperature (°F)	120	570
Design pressure (psi)	900	900
Operating pressure (psi)	900	900

a. Includes weight of pipe, contents, and insulation.

TABLE 4. PIPE DATA AND MATERIAL SPECIFICATIONS--SIS PROBLEM SI-51 TUBING

OD (inches)	0.375
Thickness (inches)	0.049
Weight ^a (lb/ft)	0.2
Material	316 SST
E_{cold} (10^6 psi)	28.3
S_h (psi)	18800
Design temperature (°F)	120
Operating temperature (°F)	120
Design pressure (psi)	900
Operating pressure (psi)	900

a. Includes weight of pipe, contents, and insulation.

TABLE 5. PIPING SYSTEM VALVE DESCRIPTIONS

Model	Valve	Weight (lb _f)	Face-to-Face Dimension (in.)
SI-51	HV851A & B	2960/2338 ^a	35
	HV852A & B	2160/2338	33
	12-600-222	1463/-	33
SI-04	862-12-C42	750/-	27.5
	HV-853A	2550/1275	33
	HV-854A	2210/1105	30
	14-300-438	1473/737	32.5
	861A-16-G42	1053/527	16
SI-158	MOV-850C	801/393	21
	867C-6-C58	259/-	17
SI-51 Tubing	870E-3/4"-T5	5/-	5
	SIS-FT-912 iso valve	9/0	2

a. The first number is the weight of the valve body and the second is the weight of the valve stem.

TABLE 6. SUMMARY OF FIRST TEN NATURAL FREQUENCIES AND PERIODS OF VIBRATION--SIS MODEL SI-51

Mode	W Rods		W/O Rods	
	<u>f</u> (Hz)	<u>T</u> (second)	<u>f</u> (Hz)	<u>T</u> (second)
1	4.739	0.211	4.455	0.225
2	5.518	0.181	5.147	0.194
3	5.794	0.173	5.794	0.173
4	7.731	0.129	6.111	0.164
5	8.298	0.121	7.645	0.131
6	8.355	0.120	8.271	0.121
7	9.127	0.110	8.380	0.119
8	9.805	0.102	8.805	0.114
9	10.697	0.094	9.465	0.106
10	11.037	0.091	10.464	0.096

TABLE 7. SUMMARY OF FIRST TEN NATURAL FREQUENCIES AND PERIODS OF VIBRATION--SIS MODEL SI-04

Mode	W Rods		W/O Rods	
	<u>f</u> (Hz)	<u>T</u> (second)	<u>f</u> (Hz)	<u>T</u> (second)
1	1.024	0.977	1.024	0.976
2	3.277	0.305	3.219	0.311
3	4.545	0.220	3.639	0.275
4	5.364	0.186	4.545	0.220
5	7.209	0.139	5.364	0.187
6	7.433	0.135	7.209	0.138
7	7.585	0.132	7.437	0.134
8	8.638	0.116	7.586	0.132
9	8.968	0.112	8.638	0.116
10	9.602	0.104	8.972	0.112

TABLE 8. SUMMARY OF FIRST TEN NATURAL FREQUENCIES AND PERIODS OF VIBRATION--SIS MODEL SI-158

<u>Mode</u>	<u>f</u> <u>(Hz)</u>	<u>T</u> <u>(second)</u>
1	6.288	0.159
2	11.141	0.090
3	13.236	0.076
4	14.010	0.071
5	15.529	0.064
6	16.197	0.062
7	18.456	0.054
8	19.290	0.052
9	20.911	0.048
10	23.716	0.042

TABLE 9. SUMMARY OF FIRST TEN NATURAL FREQUENCIES AND PERIODS OF
VIBRATION--SIS MODEL SI-51 TUBING

<u>Mode</u>	<u>f</u> <u>(Hz)</u>	<u>T</u> <u>(second)</u>
1	5.791	0.173
2	9.046	0.110
3	9.202	0.109
4	11.587	0.086
5	12.268	0.082
6	13.815	0.072
7	15.480	0.064
8	18.422	0.054
9	18.584	0.054
10	23.364	0.043

TABLE 10. COMPARISON OF STRESS RESULTS FOR S1-S1

Node/UP EG&G/Licensee	Component	SIF EG&G/Licensee	Inertial Stress: Equation 9			2(SAM) Stress: Equation 10		
			Stress (ksi)		% Difference Based on Max. Stress (62.5 ksi)	Stress (ksi)		% Difference Based on Max. Stress (17.2 ksi)
			EG&G	Impell		EG&G	Licensee	
10/7	Reducer-lrg	3.000	63.9	61.1	4	9.1	9.3	1
15/10	Reducer-sm	3.000	36.5	34.9	3	5.2	5.4	1
20/25	Straight pipe	1.000	12.9	12.2	1	1.5	1.6	1
22/30	Straight pipe	1.000	12.8	12.0	1	1.5	1.6	1
30/36	Tee-run	1.427/1.426	13.9	12.9	2	2.0	2.3	2
50/45	Tee-branch	1.427/1.426	15.4	10.6	8	3.0	3.1	1
	-North run	1.427/1.426	14.4	13.2	2	3.8	3.7	1
	-South run	1.427/1.426	12.5	12.6	0	3.2	3.2	0
55/50	Reducer-lrg	2.000	15.9	12.7	5	5.0	4.8	1
60/55	Reducer-sm	2.000	15.1	12.1	5	4.5	4.2	2
830/300	Straight pipe	1.000	11.3	9.2	3	1.8	1.7	1
80/62	Straight pipe at branch	1.500	10.5	9.5	2	2.6	2.6	0
95/70b	Elbow	1.000	8.2	7.2	2	2.0	1.9	1
105/76a	Elbow	1.000	8.6	8.0	1	1.9	1.8	1
110/76b	Elbow	1.000	8.3	8.6	0	2.3	2.4	1
145/105a	Elbow	1.000	8.8	8.8	0	4.3	4.4	1
180/115	Straight pipe	1.000	11.1	11.0	0	10.1	9.4	4
845/305	Straight pipe near valve	1.000	12.7	9.8	5	1.7	1.7	0
850/310a	Elbow	1.848	16.3	12.2	7	4.9	4.6	2
875/325a	Elbow	1.848	14.2	10.6	6	5.2	4.4	5
885/330a	Elbow	2.423/2.421	12.2	11.0	4	3.7	3.6	1
900/310a	Elbow	1.848	10.4	10.2	0	1.8	1.6	1
915/305a	Elbow	1.848	8.0	7.4	1	0.3	0.1	1
250/156	Branch, West run	2.077	13.2	11.0	4	16.5	17.8	7
700/449	Elbow	1.643	8.8	8.7	0	17.6	16.5	6
425/260	Reducer-sm	2.000	26.4	29.2	4	5.0	5.2	1
430/265	Reducer-lrg	2.000	27.0	28.9	3	5.2	5.8	3
435/350	Tee-Branch	1.427/1.426	29.9	27.5	4	5.2	7.7	14
	-North run	1.427/1.426	21.6	22.6	2	3.8	4.5	4
	-South run	1.427/1.426	35.1	30.4	8	5.3	7.4	12
450/395	Straight pipe	1.000	18.7	17.0	3	2.4	3.0	3
660/355	Tee-branch	1.427/1.426	26.8	23.8	5	2.9	4.8	11
	-South run	1.427/1.426	26.9	24.0	5	2.9	4.5	9
675/370	Elbow	1.848	15.5	15.6	0	3.0	3.5	3
680/380	Elbow	1.848	23.3	21.7	3	4.4	4.2	1
680/380	Reducer-sm	3.000	34.0	31.8	4	7.1	6.8	2
685/382	Reducer-lrg	3.000	50.4	54.8	7	12.6	11.6	6

TABLE 11. COMPARISON OF STRESS RESULTS FOR SI-04

Node/UP EG&G/L licensee	Component	SIF EG&G/L licensee	Inertial Stress; Equation 9			2(SAM) Stress; Equation 10		
			Stress (ksi)		% Difference Based on Max. Stress (23.85 ksi)	Stress (ksi)		% Difference Based on Max. Stress (11.05 ksi)
			EG&G	L licensee		EG&G	L licensee	
60/70	Straight pipe	1.000	4.4	5.1	3	0.5	0.4	1
70/80	Straight pipe	1.000	5.4	4.7	3	0.7	0.6	1
75/90	Straight pipe	1.000	4.5	3.8	3	0.6	0.6	0
85/100	Straight pipe	1.000	7.4	7.5	0	1.6	1.5	1
100/110	Straight pipe	1.000	10.4	10.5	0	0.8	0.8	0
110/120	Straight pipe	1.000	8.0	7.9	0	0.7	0.7	0
120/130	Straight pipe	1.000	12.3	12.5	1	0.6	0.6	0
130/140	Straight pipe	1.000	5.2	3.3	8	1.4	1.4	0
135/145	Elbow	3.227/3.225	11.6	7.6	17	4.8	4.9	1
140/145	Elbow	3.227/3.225	14.2	11.0	13	5.0	5.0	0
145/150	Straight pipe	1.000	7.2	5.7	6	1.6	1.6	0
150/153	Straight pipe	1.000	8.8	7.2	7	1.9	2.0	1
155/155	Elbow	3.227/3.225	19.1	15.7	15	5.8	5.5	3
160/155	Elbow	3.227/3.225	15.4	12.4	13	4.1	4.2	1
175/165	Straight pipe	1.000	23.5	16.2	31	5.5	5.4	1
190/175	Straight pipe	1.000	14.7	14.3	2	6.1	6.0	1
200/190	Straight pipe	1.000	10.5	9.2	5	1.9	1.9	0
205/200	Straight pipe	1.000	13.4	12.4	4	1.9	2.1	2
215/210	Straight pipe	1.000	6.7	7.1	2	3.9	3.8	1
220/215	Elbow	3.227/3.225	11.1	12.0	4	11.2	10.9	3
225/215	Elbow	3.227/3.225	11.9	11.6	1	8.2	7.7	5
230/220	Elbow	3.227/3.225	15.0	15.4	7	8.6	8.3	3
235/220	Elbow	3.227/3.225	14.2	12.0	9	9.1	9.2	1
250/225	Elbow	3.227/3.225	14.7	13.8	4	7.5	8.0	5
255/225	Elbow	3.227/3.225	11.7	11.2	2	6.9	7.7	7
265/230	Elbow	3.227/3.225	10.1	5.4	20	8.2	7.2	9
270/230	Elbow	3.227/3.225	9.8	5.9	16	5.6	6.8	11
285/235	Elbow	4.229/4.232	13.5	8.1	23	7.3	6.1	11
290/235	Elbow	4.229/4.232	16.8	9.2	32	6.3	6.6	3
320/245	Tee-branch	2.539/2.538	13.4	9.4	17	4.6	5.0	4
	-North run		12.5	8.9	15	4.6	5.0	4
335/247	Reducer-1rg	2.000	11.5	7.0	19	3.9	3.1	7
340/247	Reducer-sm1	2.000	10.2	7.2	13	3.3	3.6	3
346/255	Elbow	3.851/3.841	16.8	11.0	24	6.2	6.1	1
350/255	Elbow	3.851/3.841	19.0	13.1	25	6.3	5.8	5
360/260	Tee-branch	2.317/2.316	13.5	9.9	15	3.8	3.9	1
	-East run		14.2	11.7	10	3.6	4.0	4
	-North run		14.2	12.5	7	1.9	2.0	1
370/270	Reducer-1rg	2.000	19.2	24.2	21	5.0	6.9	17
375/270	Reducer-sm1	2.000	20.5	13.1	30	5.0	3.2	16
385/285	Elbow	3.851/3.841	20.9	17.0	16	3.1	2.6	5
390/285	Elbow	3.851/3.841	20.2	14.5	24	3.2	1.6	14
390/290	Elbow	3.851/3.841	20.2	14.5	24	3.2	1.6	14
395/290	Elbow	3.851/3.841	14.7	10.5	18	2.9	1.8	10

TABLE 12. COMPARISON OF STRESS RESULTS FOR SI-15B

Node/DP EG&G/L licensee	Component	SIF EG&G/L licensee	Inertial Stress: Equation 9			2(SAM) Stress: Equation 10		
			Stress (ksi)		% Difference Based on Max. Stress (13.1 ksi)	Stress (ksi)		% Difference Based on Max. Stress (6.75 ksi)
			EG&G	Licensee		EG&G	Licensee	
5/1000	Straight pipe	1.000	5.5	6.0	4	2.2	3.1	13
20/1030	Straight pipe	1.000	4.4	3.6	6	0.2	0.2	0
35/1090	Straight pipe	1.000	5.1	4.3	6	0.8	1.0	3
40/1100	Straight pipe	1.000	4.9	4.1	6	1.0	1.1	1
45/C10B	Elbow	1.643	4.4	3.8	5	0.8	0.6	3
50/C10A	Elbow	1.643	4.2	3.6	5	0.6	0.6	0
55/C9B	Elbow	1.643	4.1	3.3	6	1.0	1.5	7
60/C9A	Elbow	1.643	4.3	3.4	7	1.2	2.0	15
70/C8B	Elbow	1.643	4.2	3.4	6	1.6	1.2	6
75/C8A	Elbow	1.643	4.2	3.5	5	1.2	0.9	4
80/C7B	Elbow	1.643	4.2	3.6	5	0.8	1.0	3
85/C7A	Elbow	1.643	4.2	3.8	3	0.8	1.6	12
95/C6B	Elbow	1.643	4.3	4.0	2	1.2	1.1	1
100/C6A	Elbow	1.643	4.2	3.9	2	1.2	1.1	1
105/1270	Straight pipe	1.000	4.1	3.6	4	0.8	0.6	3
115/1290	Straight pipe	1.000	4.0	3.7	2	0.6	0.8	3
120/1320	Straight pipe	1.000	4.2	4.4	2	1.2	1.2	0
125/C5B	Elbow	1.000	4.3	4.4	1	1.2	1.2	0
130/C5A	Elbow	1.000	4.5	4.4	1	1.4	1.3	1
135/1360	Straight pipe	1.000	4.6	4.8	2	2.0	1.7	4
140/1350	Run @ branch	1.500/2,100 ^a	5.2	6.2	8	3.4	4.2	12
145/C4B	Elbow	1.643	5.6	5.7	1	4.0	3.6	6
150/C4A	Elbow	1.643	5.9	6.4	4	4.6	4.2	6
155/1430	Straight pipe	1.000	5.5	6.2	5	3.0	2.6	6
160/1440	Straight pipe	1.000	5.5	5.9	3	3.0	2.6	6
165/1460	Straight pipe	1.000	4.9	5.3	3	1.8	1.9	1
195/C3B	Elbow	1.643	6.2	7.8	12	8.1	6.8	19
200/C3A	Elbow	1.643	5.7	8.1	18	7.0	6.5	7
205/V2C	Straight pipe	1.000	5.7	7.5	14	3.8	3.7	1
230/1700	Elbow	1.099	9.2	10.6	11	3.8	2.7	16
235/1704	Elbow	1.099	9.6	10.9	10	4.0	2.8	18
235/1705	Elbow	1.099	9.6	11.3	13	4.0	2.9	16
240/1706	Elbow	1.099	10.0	12.5	19	4.6	3.2	21
250/1717	Straight pipe	1.000	12.7	13.5	6	4.0	2.8	18

a. The EG&G analyst assumed a full penetration weld in place below the fillet weld per common nuclear grade weld practice. The licensee's analyst conservatively assumed that only the visible weld could be credited. This led to the difference in SIF.

TABLE 13. COMPARISON OF STRESS RATIO RESULTS FOR SI-51 SUPPORTS

Support ID	Type	Node/DP	Component	Stress Ratio		Notes
				EG&G	Licensee	
SI-05-0320-H013	snubber	22/30	anchor bolt	1.6	--	1
SI-05-6005-H006	strut	260/157	clamp	0.3	0.9	2
SI-02-6005-H005	snubber	375/225	snubber	1.8	1.8	3
SI-06-6005-H004	snubber	380/230	clamp	1.8	1.0	4
SI-06-0319-H007	snubber	485/537	anchor bolt	2.0	0.9	5
SI-05-0320-H017	spring hanger	855/310	spring hanger	0.86	0.87	
SI-05-0320-H011	snubber	861-315	anchor bolt	0.64	0.96	6

1. The supports added to SI-51 made the snubber unnecessary and it was removed.

2. Based on the shape and dimensions recorded in the field, the EG&G analyst concluded the clamp was part of the snubber assembly originally installed. The licensee's analyst made note of the discrepancy, but used the load rating for the clamp associated with the strut assembly that replaced the snubber. This was conservative.

3. The support had been turned over to the licensee's design group for modification to increase load capacity.

4. The EG&G analyst used the published catalog load rating. The licensee's analyst contacted the manufacturer and obtained an increased load rating for the clamp.

5. Supports added to the piping in the vicinity of this support reduced the loads on it to acceptable levels.

6. Supports added to the piping in the vicinity of this support increased its loads.

TABLE 14. COMPARISON OF STRESS RATIO RESULTS FOR SI-158 SUPPORTS

Support ID	Type	Node/DP	Component	Stress Ratio		Notes
				EG&G	Licensee	
SI-01-6007-H005	strut	35/1090	clamp	0.23	0.21	
SI-01-6007-H30I	guide	105/1270	baseplate	0.70	0.96	1
SI-01-6007-H00E	snubber	155/1430	baseplate	0.33	0.34	
SI-01-6007-H00D	snubber	160/1440	baseplate	0.52	0.39	2
SI-01-6007-H000	snubber	165/1460	anchor bolts	0.29	0.93	3
SI-01-6007-H00B	spring hanger	180/1500	spring hanger	0.78	0.79	
SI-01-6007-H00A	spring hanger	240/1710	spring hanger	0.72	0.58	4

1. The EG&G analyst did a finite element analysis, the licensee's analyst a simple beam calculation. Both are acceptable.

2. The EG&G analyst did a simple beam calculation, the licensee's analyst a finite element analysis. Both are acceptable.

3. The licensee's analysis was based on factoring the results of a previous analysis to account for different loads. In the previous analysis support loads had been increased to account for the possibility of support dynamic response. Current criteria ensure support rigid response, so that support loads need not be increased.

4. Different hot loads were used. EG&G's hot load was based on design drawings, because the hot load hanger setting was not recorded during walkdown. The licensee's load was based on the current setting.

APPENDIX A

DESCRIPTION OF COMPUTER CODE

C

APPENDIX A

COMPUTER PROGRAM DESCRIPTION

The NUPIPE-II computer program performs linear elastic analysis of three-dimensional piping systems subject to thermal, deadweight, seismic, and other static and dynamic loads. The NUPIPE-II program is also designed to perform stress and fatigue analyses in accordance with the ASME Boiler and Pressure Vessel Code, Section III, Nuclear Power Plant Components, 1974 Edition through the Summer 1975 Addenda; and the ANSI B31.1 Code, 1967 and Summer 1973 versions. NUPIPE-II may also be utilized to assure compliance with later piping code requirements provided the analyst takes into consideration any possible changes. Piping systems of more than one classification can be analyzed.

NUPIPE-II utilizes the finite element method of analysis with special features incorporated to accommodate specific requirements of piping system analysis. In accordance with the finite element method, the continuous piping is mathematically idealized as an assembly of elastic structural members connecting discrete nodal points. Nodal points are placed in such a manner as to isolate particular types of piping elements, such as straight runs of pipe, elbows, valves, etc., for which force-deformation characteristics can be categorized. Nodal points are also placed at all discontinuities, such as piping supports, concentrated weights, branch lines, and changes in cross-section. System loads such as weights, equivalent thermal forces, and earthquake inertia forces are applied at the nodal points. For the deadweight and dynamic time-history and response spectra analyses, distributed weight properties of the piping as well as concentrated weights, such as valves, pumps, or snubbers, can be considered. A lumped mass model of the piping system is used for all dynamic analyses. Both translational and rotational degrees-of-freedom may be considered.

For further information concerning NUPIPE-II capabilities or analytical procedures, contact Applied Mechanics Branch of EG&G Idano, Inc.

APPENDIX B

MICROFICHE OF COMPUTER OUTPUT .

NOTE: MICROFICHE OF COMPUTER OUTPUT
(PAGES B-2 TO 5) ARE AVAILABLE
IN THE PUBLIC DOCUMENT ROOM and
LOCAL PUBLIC DOCUMENT ROOM

From Page B2 TO

B5 - are

Microfiche

Egn 9 calculation

$$\begin{aligned} 1) M_a &= [M_x^2 + M_y^2 + M_z^2]^{1/2} && (\text{ult moment}) \\ 2) M_b &= [M_x^2 + M_y^2 + M_z^2]^{1/2} && (\text{service moment}) \\ 3) \text{Seismic stress} &= \frac{0.75 (IF) (M_a + M_b)}{(1000) \text{ in}^2} \end{aligned}$$

$$4) \text{Total} = \text{pressure} + \text{seismic}$$

Egn 10 calculation

$$\begin{aligned} 1) M &= [M_x^2 + M_y^2 + M_z^2]^{1/2} \\ 2) \text{stress} &= \frac{(IF) M}{(1000) \text{ in}^2} \end{aligned}$$

Calculation of stresses where NUPISE
IF couldn't be specified correctly.

Run AF08, 8/5/64

ST-51 Reducer Egn 10 stress Calc.

Node	Po	to	My	My 1 M2	M	SIF	Z	stress
10	10.75	0.543	111672	78174	173549	3.0	45.54	11.4
15	12.75	0.687	120461	67557	165449	3.0	74.6	6.5
35	12.75	0.687	55535	112010	193806	2.0	74.6	5.2
60	14.00	0.580	55535	113803	176591	2.0	78.77	4.5
423	11.00	0.580	156074	81074	207938	2.0	78.77	5.3
412	12.75	0.687	156074	102013	215541	2.0	74.6	6.8
480	12.75	0.687	74538	160767	193080	3.0	74.6	7.8
455	10.75	0.543	91426	173847	206670	3.0	45.54	13.6

PI-51 Reducer Egn 7 Stress Calc

D, in	L, in	t _m	Weight			MHE			SIF	Z	Stress				
			M _x	M _y	M _z	M _x	M _y	M _z			pressure	seismic	Total		
10	10.75	0.575	912.43	1136.2	411066	100701	688256	889946	267671	1123454	3.0	45.54	6.12	60.48	66.6
15	12.75	0.607	15473	15565	41066	90800	578214	73076	267761	967085	1.0	74.6	6.24	31.97	38.2
55	12.75	0.574	1667	28636	23390	37012	183875	268082	281021	423544	2.0	74.6	6.26	9.26	15.5
60	14.00	0.580	1667	25719	10605	278710	183875	160719	279413	371105	2.0	78.79	8.15	7.60	15.7
40	14.00	0.580	307	11914	69274	76292	240182	487463	456774	709962	2.0	78.79	8.15	14.85	23.0
410	12.75	0.587	307	12268	120716	121398	240182	442524	460810	682631	2.0	74.6	6.26	16.17	22.4
620	12.75	0.687	18100	8113	18443	27119	325042	776079	256654	879695	3.0	74.6	6.26	27.35	33.6
685	12.75	0.572	14579	11013	18443	26108	326425	864564	256654	957303	3.0	45.54	6.12	48.58	54.7
2000															

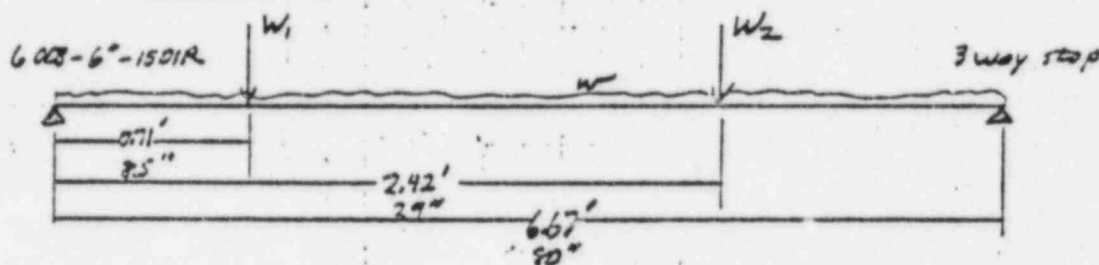
APPENDIX C

SMALL-BORE PIPING HAND CALCULATION

A small bore piping calculation will be done for the SI-51 tubing stress results will be compared to those for a finite element analysis of the piping.

By inspection, the critical span is the terminating span at the connection to 6008-6"-1501R.

N-S Dir



W_1 - small hand valve - assume 5 #

W_2 - out of plane span, 1.17' long

$$W = (0.2 \text{ #/ft}) (1.17') = 0.23 \text{ #}$$

Span properties; 3/8" 35T tubing

$$D_o = \frac{3}{8}'' = 0.375''$$

$$t_m = 0.049''$$

$$I = \frac{1}{4} \pi \left[\left(\frac{0.375}{2} \right)^4 - \left(\frac{0.375}{2} - 0.049 \right)^4 \right]$$

$$I = 6.817 \cdot 10^{-4} \text{ in}^4$$

$$S = 2(6.817 \cdot 10^{-4}) / 0.375 = 3.676 \cdot 10^{-3} \text{ in}^3$$

acceleration -

$$a_{ms}/peak = 2.196 g$$

eg. wt: $W = (0.2 \text{ #/ft}) (1 \text{ ft}/12 \text{ in}) (2.196 + 1 g) = 0.053 \text{ #/in}$

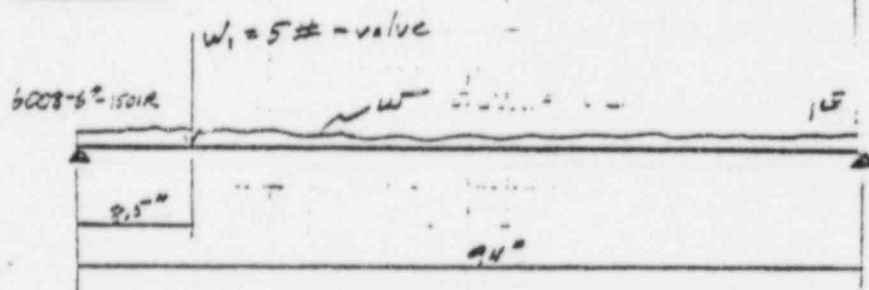
$$W_1 = (5) (3.196) = 15.980$$

$$W_2 = (.23) (3.196) = 0.735$$

Two HP41C programs (see attachment) are used to solve the simply supported beam for:

W1:			W2:			W2:		
LENGTH?	0.0000	RUN	L=?	38.0000	RUN	L=?	38.0000	RUN
MODULUS?	30.0000	RUN	A=?	3.5000	RUN	A=?	30.0000	RUN
X?	3.636-83	RUN	F=?	15.9380	RUN	F=?	29.0000	RUN
FB=	4.429.4211	RUN	G=?	3.636-83	RUN	G=?	.7358	RUN
X?	8.5000	RUN	R1=	14.2921	RUN	R1=	0.4666	RUN
FB=	10.779.2994	RUN	R2=	1.6979	RUN	R2=	0.2664	RUN
X?	29.0000	RUN	GA=	33.387.3868	RUN	GA=	3.737.1537	RUN
FB=	11.161.1611	RUN	X=?	29.0000	RUN	X=?	8.5000	RUN
			GX=	23.815.8784	RUN	GX=	1.895.3744	RUN
			X=?	48.0000	RUN	X=?	48.0000	RUN
			GX=	18.673.4422	RUN	GX=	1.931.1611	RUN

E-W Dir



acceleration:

$$-a_{z-w}/max = 1.934 g$$

equivalent wt:

$$W = (0.2 \text{ #/ft}) (1 \text{ ft}/12 \text{ in}) (1.934 g)$$

$$W = 0.033 \text{ #/in}$$

$$W_1 = (5) (1.934) = 9.670 \text{ #}$$

C-3

using the same solution scheme:

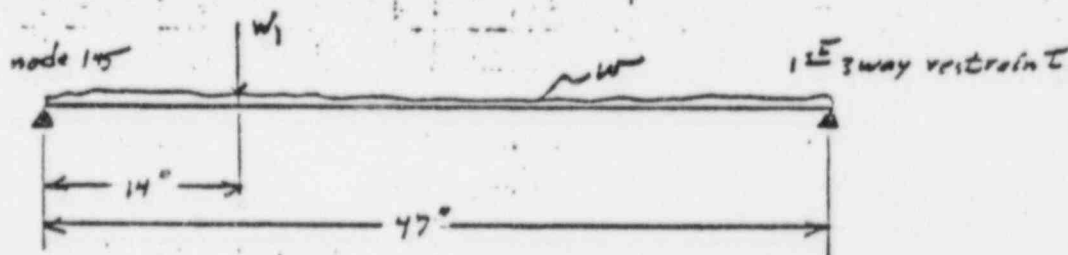
w_1 :

XEQ *D99	
WEIGHT/IN?	.0330 RUN
LENGTH?	94.0000 RUN
MODULUS?	3.636-03 RUN
X?	8.5000 RUN
FB= 3,297.9579	RUN
X?	9.0000 CLX
	47.0000 RUN
FB= 10,024.3399	

w_2 :

XEQ *D99	
L=?	94.0000 RUN
A=?	8.5000 RUN
F=?	7.0000 CLX
	.0000 CLX
	9.9200 RUN
S=?	3.636-03 RUN
R1= 9.0230	RUN
R2= 0.0970	RUN
SA= 21,093.3221	RUN
X=?	47.0000 RUN
SM= 11,000.0000	

Vertical Dir



acceleration:

$$a_{v/\max} = 1.698$$

equivalent w_1 :

$$w = (0.2)(1/2)(1.698) = 0.0293 \text{ #/in}$$

$$w_1 = (0.2)(1.5)(1.698) = 0.509 \text{ #}$$

30N692

5/30/86

SB Piping Conformatory

M/R P4

Using the same solution scheme:

w:			w ₁ :		
WEIGHT/IN?	KE1 088		L=?	KE0 088	
LENGTH?	.8233	RUN	A=?	47.8888	RUN
MODULUS?	47.8888	RUN	F=?	14.8888	RUN
X?	3.636-83	RUN	S=?	.5898	RUN
FB= 1.797.9373	14.8888	RUN	R1= 0.3574	3.636-83	RUN
X?	23.5888	RUN	R2= 0.1516		RUN
SA= 2.149.1577			SA= 1.376.8621		RUN
			X=?		
				23.5888	RUN
			SA= 177.9270		

Load combinationsN-S Dir

Location	2.5	29.0	40.0
w	4429	10799	11661
w ₁ (w ₂)	3388	27815	18678
w ₂	342	3737	2931
Total	38159	38351	32270
		limit	

E-W Dir

Location	2.5	47.0
w	7298	10024
w ₁	21043	11595
Total	24341	21619
	limit	

C-5

Vertical Der.

Location	14.0	23.5
W	1798	2149
W ₁	1376	180
total	3174	3129
limits		

Combined stress

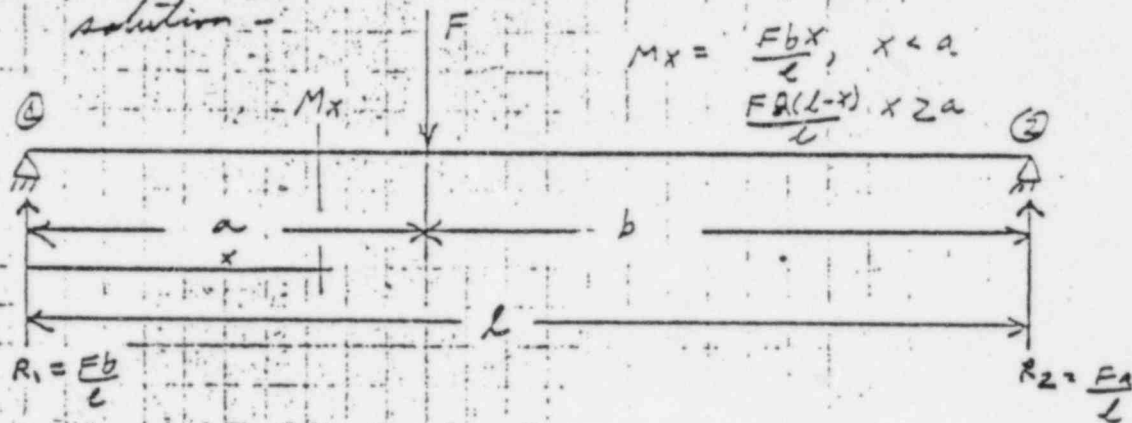
IF : For socket welded fittings $i = 2.1$

$$\sigma = (0.75)(2.1) [38351^2 + 24311^2 + 3174^2]^{1/2}$$

$$\sigma = 71758 \text{ psi N.G.}$$

The piping is cold, so no thermal stress calculation was required.

55 solution -



$$M_x = \frac{Fbx}{L}, \quad x < a$$

$$\frac{Fa(L-x)}{L}, \quad x > a$$

given: L, a, F, I, S

needed: $R_1, R_2, F_b(x)$

Register

Controls

0
1
2
3
4
5
6

L
a
F
S
b
X

PRP *SS*

01 *LBL *SS*
02 XEQ *INP*
03 RCL 02
04 RCL 05
05 *
06 RCL 00
07 /
08 *R1 = *
09 ARCL X
10 PROMPT
11 RCL 02
12 RCL 01
13 *
14 RCL 00
15 /
16 *R2 = *
17 ARCL X
18 PROMPT
19 RCL 02
20 RCL 05
21 *
22 RCL 01
23 *
24 RCL 00
25 /
26 RCL 04
27 /
28 *SA = *
29 ARCL X
30 PROMPT
31 SF 01

32 *LBL 01
33 * X=?
34 PROMPT
35 STO 06
36 RCL 01
37 X>Y?
38 GTO 02
39 RCL 00
40 RCL 06
41 -
42 RCL 02
43 *
44 RCL 01
45 *
46 RCL 00
47 /
48 GTO 03
49 *LBL 02
50 RCL 02
51 RCL 05
52 *
53 RCL 06
54 *
55 RCL 00
56 /
57 *LBL 03
58 RCL 04
59 /
60 *SX = *
61 ARCL X
62 PROMPT
63 GTO 01
64 END

PRP *INP*

01 *LBL *INP*
02 * L=?*
03 PROMPT
04 STO 00
05 * A=?*
06 PROMPT
07 STO 01
08 * F=?*
09 PROMPT
10 STO 02
11 * S=?*
12 PROMPT
13 STO 04
14 RCL 00
15 RCL 01
16 -
17 STO 05
18 END

KCLD1
RCLD0

DSS - Moment calculation for uniformly loaded SS beam.



$$M(x) = \frac{wx}{2} (L-x)$$

$$F_b = \frac{M(x)}{S}$$

```

01+LBL "DSS"
02 "WEIGHT/IN?" ← unit
03 PROMPT      wt.
04 STO 01      )?
05 "LENGTH?" ← beam
06 PROMPT      length
07 STO 02      3
08 "MODULUS?" ← S (units T)
09 PROMPT
10 STO 03
11+LBL 01
12 "X?" ← X
13 PROMPT
14 STO 04
15 RCL 01
16 *
17 2
18 /
19 RCL 02
20 RCL 04
21 -
22 *
23 RCL 03
24 /
25 "FB=" → Fb(x)
26 RCL X
27 PROMPT
28 GTO 01
    
```

APPENDIX D

PIPING SUPPORT CALCULATIONS

INCL 4/11/82

SON/SSA Letter. Anal

NAIR R2

Support summary sheet for C-51

Support	Node/O.P	Stress Ratio	Page
SI-05-0320-H013	22/30	1.60	D-5
-H014	25/75	0.167	D-5
SI-05-6004-H010	85/65	0.86	D-5
-H012	100/75	0.22	D-6-7
-H007	115/81	0.09	D-7-9
-H006	120/95	—	D-9
-H001	125/90	0.15	D-9-10
-H014	135/95	0.50	D-10-11
-H003	145/100	—	D-12
-H005	170/110	0.57	D-12-13
SI-02-6004-H002	180/215	0.69	D-13-14
-H006	195/120	0.79	D-15
-H001	200/125	0.77	D-16
-H003	235/145	0.76	D-16-19
SI-02-6005-H006	260/157	0.33	D-20
-H003	275/165	0.15	D-20-21
-H005	285/175	0.47	D-22
-H001	290/177	0.67	D-22-24
-H008	300/196	0.19	D-25-26
-H002	315/195	0.80	D-27
-H004	330/205	0.44	D-28
	340/210	—	D-29
	345/215	—	D-30
SI-06-6005-H005	375/225	1.82	D-30
-H004	380/230	1.78	D-30-31
-H003	405/250	0.79	D-32-34
SI-06-0319-H004	445/390	0.78	D-35
-H007	485/537	1.98	D-35-37
-H006	490/577	0.81	D-38
SI-06-0323-H013	520/560	—	D-39
-H005	675/375	0.86	D-39
SI-02-6008-H001	710/455	0.06	D-39
-H002	720/460	0.05	D-40-42
SI-02-6006-H001	755/425	0.17	D-43
-H002	765/430	0.06	D-43
SI-02-6007-H001	800/405	0.21	D-43
-H002	810/410	0.05	D-43
	840/—	—	D-43
SI-05-0720-H017	855/310	9.37	D-43
-H016	855/312	0.63	D-44-46
-H011	861/315	0.66	D-46
-H004	870/320	0.17	D-47

Support Summary Chart For 251 (19-26)

	Page
Loads	D-48-53
Standard component tables	
Lugs	D-55-59
Spring hangers	D-60
Sway struts	D-61-63
Snubbers	D-64-65
HP-41C Programs -	D-54, D-66

Support Summary Sheet for SI-158

Support	Node/D.P.	Stress Ratio	Page
SI-01-6007-H00L	20/1030	0.17	D-156-159
--H00K	35/1090	0.47	D-153-155
-H00J	40/1100	0.28	D-150-152
-H30I	105/1270	0.70	D-107-149
-H30H	115/1290	0.84	D-95-106
-H30G	120/1320	0.20	D-94
-H30F	135/1360	0.34	D-85-93
-H00E	155/1430	0.33	D-83-84
-H00D	160/1440	0.52	D-80-82
-H00C	165/1460	0.88	D-77-79
-H00B	180/1500	0.78	D-72-76
-H00A	240/1710	0.72	D-67-71

Loads

D-160-

Node 22 / d.p. 70 / SI-05-0720-H013 / Snubber

Based on a review of the design and the calculations in the FCR, this support design is controlled by the anchor bolt capacity. As a first pass, assume that the bolt closest to the snubber $\bar{c}$ carries the full load. (9193#),

The bolt is a 1" Super Kwik, @ 6 1/2" embedment.

$$\bar{F}_a = 22945 / 4 = 5737$$

$$\bar{F}_b = 19413 / 4 = 4853$$

$$\left(\frac{9193}{5737} \right) = 1.60 \text{ NR}$$

The reduced factor of safety cannot be applied because all bolts are in tension. A detailed analysis would not show sufficient load redistribution to pass the design.

Node 25 / d.p. 75 / SI-05-0720-H014 / Spring hanger

Per the spring hanger Table, $u = 0.67$

Node 85 / d.p. 65 / SI-05-6594-H010 / Spring Hanger

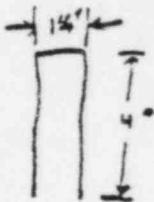
Per the spring hanger Table, $u = 0.86$

Note 100/dp 75 / SE-05-6004-H012 / 5 subtest

The scumblers from near bracket to pipe clamps, have been qualified in the scumbler table. This leaves:

- lugs attach weld
- bracket attach weld

a) lugs: the lugs are $1\frac{1}{4}" \times 4"$ on the pipe surface, 4 total, with a $\frac{1}{4}"$ fillet weld as shown:



Area of weld:

$$A = \frac{1}{4} (1\frac{1}{4} \times 2.4)$$

$$A = 2.317 \text{ in}^2$$

the load is at an angle, with the following direction cosines:

$$\text{Scumbler: } \vec{S} = (.7024, .6175, -.7244)$$

$$\text{pipe: } \vec{P} = (0, 1.0, 0)$$

The axial load is:

$$F_{\text{axial}} = F_{\text{SD}} (\vec{S} \cdot \vec{P})$$

$$F_{\text{axial}} = 5028 (.6175) = 3085.16$$

$$F_{\text{trans}} = \sqrt{F_{\text{SD}}^2 - F_{\text{axial}}^2} = 3971.16$$

Per the lug calculation, $u = 0.22$ OK

6. Bracket attachment weld

The weld pattern is a 5" sq. with $\frac{1}{2}$ " fillet

$$A = \frac{1}{2} (5)(4) = 10 \text{ in}^2$$

Since the column is 11" the pipe,

$$F_a = F_{\text{allow}} / 10 \text{ in}^2 = 738 \text{ psi OK}$$

$$F_y = F_{\text{allow}} / 10 \text{ in}^2 = 577 \text{ psi OK}$$

Node 115/dp. 81/ IL-05-6004-H007 / guide

There are six elements to be analyzed

- 1) The horizontal W6X15 X 1'2 1/4" beam
- 2) The attachment weld to the vertical beam
- 3) The vertical W6X15 X 2'7" beam
- 4) The attachment weld of the vertical beam to the W10X24 beam running N-S (secondary steel)
- 5) The N-S W10X24 beam (secondary steel)
- 6) Main beam EHP-1923

see end of
Appendix for
listing of
the program

1) Horizontal beam -

The load is 16.9 K vertical, applied mid-span.

$$L = 1'2 \frac{1}{4}" = 14.13" \quad \text{area} = 4.43$$

$$a = L/2 = 7.07"$$

$$F = 16.9$$

$$S = 9.72$$

2 K12 stresses are acceptable - OK

The shear is

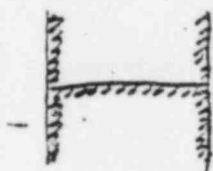
$$F_v = 5.61 / 4.43 = 1.27 \text{ ksi} - \text{OK}$$

Node 115

9/

2.) Attachment weld, horizontal to vertical beam.

The weld pattern:



$\frac{1}{4}$ " fillet weld; This compares to $t_w = \frac{1}{4}$, $t_F = \frac{1}{4}$ ". Since the weld section is nearly identical to the beam section, and the beam stresses were so low, the weld is acceptable based on comparison with the beam.

3) Vertical beam

The load is a horizontal 9.3K applied $31'11'' - 30'6'' = 1'7''$ from the top. The span is $2'7'' - 3'' = 28''$. This is in combination with an axial load of $5.61K$.

	XEQ	FF
L=?		
A=?	28.00	RUN
F=?	17.00	RUN
S=?	6.78	RUN
	9.72	RUN
R1= 2.29		RUN
R2= 4.41		RUN
S1= .91		RUN
SA= 2.76		RUN
SL= -2.80		RUN
X=?		RUN

$$L = 28$$

$$a = 17$$

$$F = 6.7$$

$$S = 9.72$$

The fixed-fixed assumption gives a maximum bending stress of 2.8 KSI and a maximum shear of $4.41 / 4.43 = 1.0$ KSI. The axial stress is $5.61 / 4.43 = 1.27$ KSI.

The beam is acceptable.

4) Attachment weld, vertical beam to secondary steel.

The attachment weld is identical to that above, and the applied loads are as low. The weld is acceptable.

Node 115

5) The W-10x29 N-S secondary steel.

Since the C of the support is within 2' of the main steel (EHP-323), local stresses are the only concern. The installation of gusset plates to support the flanges of this beam alleviates any concern in this area - acceptable.

Node 120 / dp. 95 / SI-05-6004-H006 / rod hanger

Rod hangers were given no credit for seismic analysis (piping done with & w/o rod hangers).

The are not in the scope of the support analysis.

Node 125 / dp. 90 / SI-05-6004-H001 / lateral

There are 5 elements to be analyzed:

- 1) The vertical C5x6.7x1'2 1/2" beams contacting the pipe
- 2) The attachment welds of these beams to the next element (3)
- 3) The horizontal C5x6.7x3'2 3/4" beams attaching the vertical beams (2) to the column (5)
- 4) The attachment welds of item (3) to column E-3

1) The vertical C beams will be analyzed as fixed fixed with a mid span load

L=2
14.58
20
L=2
7.25
20
S=2
6.18
20
S=2
3.46
20

$$L = 1' 2 \frac{1}{2}" = 14.5"$$

$$a = L/2 = 7.25"$$

$$F = 6.1 \text{ ksi}$$

$$S = 3 \text{ in}^2 \quad (\text{Table, EHP-30, 2nd, p 1-77})$$

$$\text{Area} = 1.97 \text{ in}^2$$

The bending stress, 37 ksi, is acceptable.

$$F_b = 3.1 / 1.97 = 1.6 \text{ ksi, ok}$$

Node 125

2) The attachment weld is a $1/4$ fillet on the inside of the C section, which has $t_w = 7/16"$ $t_f = 5/16"$. Based on the low stresses calculated for the vertical C beams and the similarity of sections here and there, the weld is acceptable. Deficiencies noted in NCR-501-R-4229 do not affect this conclusion.

3) The horizontal C beams will be analyzed as a column. They are sufficiently short that buckling need not be considered.

$$F_u = F_a = 3.25 / 1.97 = 1.62 \text{ ksi, OK.}$$

4) The attachment weld to the column consists of two $1/4"$ fillet welds $12.5"$ long loaded in shear.

$$A = (1/4)(2)(12.5) = 6.25 \text{ in}^2$$

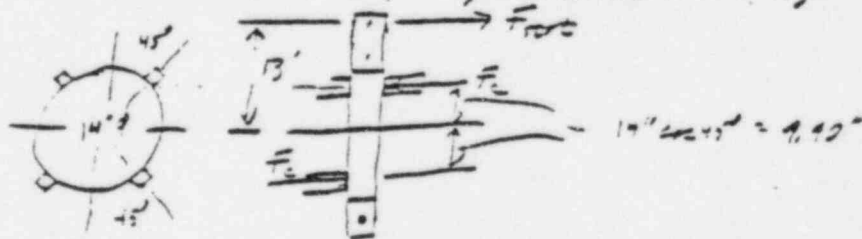
$$F_u = 6.1 / 6.25 = 0.9 \text{ ksi OK.}$$

Node 175 / dp. 95 / SI-05-6004-H014 / axial strut

There are 4 elements to be analyzed:

- 1) legs on the pipe
- 2) The strut axis
- 3) Attachment weld of legs to column E-7

2) The legs are loaded by the couple needed to resist the offset strut load, in addition to the strut load. The couple is obtained by



Node 175 from the piping analysis:

$$\overline{DCOS}_{pipe} = (0, 0, 1)$$

$$\overline{DCOS}_{stout} = (.102, 0.000, -.995)$$

The stout load is 12.5 K

$$\therefore F_{scr} = (.995)(12.5) = 12.4 \text{ K}$$

The load $\perp$ to the pipe, $\overline{F}_\perp$, is

$$\overline{F}_\perp = (.102)(12.5) = 1.3 \text{ K}$$

A moment equation gives:

$$2(9.90) \overline{F}_c = (9.9 + 16) F_{scr}$$

$$\overline{F}_c = \frac{(9.9 + 16)}{2(9.9)} 12.4 = 16.3 \text{ K}$$

The total load on the leg is

$$\overline{F}_c + \overline{F}_{scr} = 12.4 + 16.3 = 28.7 \text{ K}$$

Per the leg calculation $u = 0.50$ OK

2) Stout Assy.

Per the stout table $u = 0.42$ OK

3) The attachment weld is a $5/8"$ weld in a $75/8 \times 5"$ rectangular pattern. It is subjected to a 12.4 K normal and a 1.3 K shear load.

$$A = 2(5/8) [75/8 + 5] = 15.8 \text{ in}^2$$

$$\overline{F}_\perp = 12.4 / 15.8 = 0.8 \text{ K/in}$$

$$\overline{F}_\parallel = 1.3 / 15.8 = 0.1 \text{ K/in}, \text{ OK.}$$

Node 145 / dp 100 / SI-05-6004-H003 / Rod Hanger

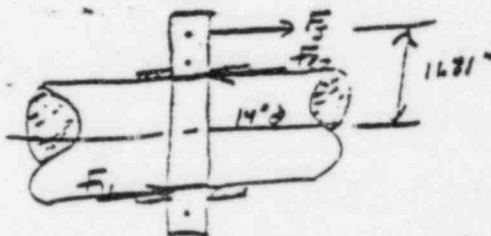
This support has been upgraded to full vertical restraint. The new design is not available for analysis.

Node 170 / dp 110 / SI-02-6004-H005 / Snubber

There are 3 elements needing analysis:

- 1) Lugs on the pipe
- 2) The snubber assembly
- 3) attachment weld of snubber assembly to col. E-5
- 4) Attachment lugs.

Based on the photographs, the lugs will be assumed to share the load and be configured as shown



$$7^{\circ} \bar{F}_{L2} = 11.81^{\circ} F_2$$

$$F_{L2} = \frac{11.81}{7} (10.2) = 17.2 \text{ K}$$

$$F_{L1} = \bar{F}_{L2} + F_2 = 10.2 + 17.2 = 27.4 \text{ K}$$

Per the lug table, $u = 0.37$ OK

2) Snubber - Per the snubber table $u = 0.57$

Node 170

3) Attachment weld of scumblers ass'y

From the ITT Grinnell Catalog P481, the bracket on a 201 shock suppressor has dimensions of $4\frac{1}{2} \times 3"$. (Note that the bracket was left when the PS-10 was substituted.

$$P = 10.2K$$

$$A = 2(4\frac{1}{2} + 3) \frac{1}{4} = 3.8 \text{ in}^2$$

$$F_a = 10.2 / 3.8 = 2.7 \text{ KSI OK. (L 27.9)}$$

Node 170 / DP 115 / SI-02-6000-14002 / 100% Strut

There are 5 elements needing evaluation:

- 1) The strut ass'y (which is evaluated in the strut table)
- 2) The attachment weld, strut ass'y to plate
- 3) The attachment weld, plate to tube steel
- 4) The $4 \times 4 \times 1/4 \times 11"$ tube steel
- 5) The attachment weld, tube steel to W18x45 beam

1) strut ass'y - per the strut ass'y table:

$$u = 0.69$$

Node 130

- 2) the attachment weld, strut ass'y to plate

this weld is 2 parallel $1/4"$ throat by $4\frac{1}{2}"$ welds

$$F = 10.6$$

$$A = 2(1/4)(4\frac{1}{2}) = 2.3 \text{ in}^2$$

$$F_a = 10.6 / 2.3 = 4.6 \text{ ksi OK}$$

- 3) the attachment weld, plate to tube

this weld pattern is 2 parallel $1/4$ fillet welds,
4" long

$$F = 10.6 \text{ K}$$

$$A = 2(1/4)(4) = 2 \text{ in}^2$$

$$F_a = 10.6 / 2 = 5.3 \text{ ksi}$$

$$F_a = 27.9 \text{ ksi} - u = 0.22 \text{ OK}$$

- 4) Tube steel.

Since the tube steel is too short for concern about buckling, its area, 7.59 in^2 , in comparison to the weld area ensures its adequacy. OK

- 5) attachment weld, tube steel to $W18 \times 45$.

This weld is identical to the weld at the other end of the tube, and is adequate by comparison OK

Node 195 / dp 120 / Σ -02-6004-H006 / H006. start

Three elements need analysis:

- 1) the start assembly
- 2) the attachment weld to Column B-6

1) The Start Assy -

Per the start ass'y table $\alpha = 0.79$

2) The attachment weld, start ass'y to col. B-6

For a 211, size 2 start ass'y, the base plate is $4\frac{1}{2}'' \times 3''$. For a $\frac{3}{8}''$ fillet weld, all around,

$$A = \frac{3}{8} (2) (4\frac{1}{2} + 3) = 5.63 \text{ in}^2$$

$$P = 9.2 \text{ K}$$

$$F_y = 9.2 / 5.63 = 1.6 \text{ K/in} \quad \text{OK}$$

Node 200 / dp 125 / 7E-02-6004-H001 / 5 strut

This support is identical to 7E-02-6004-H002 and will be qualified by comparison. The max. load is 11.8K, vs 10.6K. The controlling component for H002 is the strut, with $F/F = 0.69$. for this strut

$$F/F = (0.69) \frac{(11.8)}{(10.6)} = 0.77 < 1.0 \text{ OK}$$

Node 235 / dp 145 / 7E-02-6004-H003 / vertical guide

There are 7 points to be analyzed:

- 1) the bumper column - W 4x13x10 1/16" beam (E)
- 2) the double C 7x4.1, C-C 1'8" beam (54)
- 3) the std. components: 7/8" nut, #140, 7/8" rod (12) eye nuts (#290, part 7), #66, welded beam attach (2), #3, the washer plate (item 60)
- 4) the W 8x15x5 1/4" beam (Item B; envelope Item A and eye nuts)
- 5) attach weld for 44x4x2 1/2"x4"
- 6) 44x4x2 1/2"x4" (Item C)
- 7) attach weld, item C to support steel
- 8) attach weld, beam attach to beam

1) W 4x13x10 1/16" beam

This beam is too short to buckle.

$$A = 7.8 \text{ in}^2 \quad (\text{AISC, 8th})$$

$$F_a = 7.0 / 7.8 = 0.896, \text{ negligible OK}$$

2) double C 7x4.1, CC 1'8" beam

L=?

20.000 RUN

A=?

10.000 RUN

F=?

5.000 RUN

S=?

2.200 RUN

$$L = 20"$$

$$A = 10"$$

$$F = 5.8 \text{ K}$$

$$S = 2 \times (1.1) = 2.2 \quad (\text{AISC, 8th})$$

Node 235

$$F = 17.1 \text{ ksi} < 40.6$$

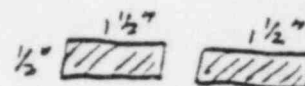
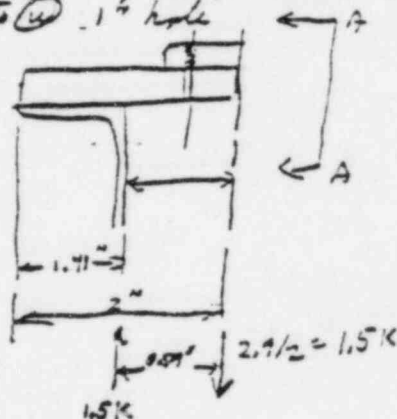
OK - 0.32

3) Standard components - load is 2.9 K

item #	catalog #	descr	rating	
25	-	7/8" nuts	quant. by rod	
12	140	7/8" rod	3.8	OK 0.76
7	290	eye nuts	3.8	OK 0.76
1	66	welded beam attach	3.8	OK 0.76

Catalog is ITT Grinnell Catalog PH 71

The washer plate, has no load rating; its a 4x4x1/2 plate @ 1" hole



sect AA

$$I = \frac{1}{12} b h^3$$

$$I = 2 \left[\frac{1}{12} (1\frac{1}{2}) (\frac{1}{2})^3 \right]$$

$$I = 0.031$$

$$F_b = \frac{[(0.59)(1.5)] [1/2 (1/2)]}{0.031} = 7.1 \text{ ksi} < 40.6$$

OK - 0.17

Perhaps an overkill, but remember the light - Remy skynog in K.C.

4) W 8 x 15 x 5' 9" beam

L = 1	69.000	RUN	L = 69"
a = 1	34.500	RUN	a = 69/2 = 34.5 (centerline)
F = 1	2.900	RUN	F = 2.9
S = 1	11.800	RUN	S = 11.8 (AISC 5th)

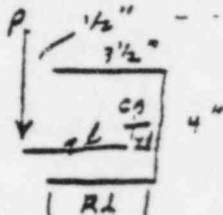
$$F = 4.2 \text{ ksi} < 40.6 \quad \text{OK} - 0.10$$

$$R_1 = 1.450$$

$$R_2 = 1.450$$

Node 2D

5) Attach weld, beam to clip



$$t = 1/4"$$

from p4-80, AISC, 8th

$$D = \frac{P}{C C_L L}$$

D = min reqd throat, 15th

P = applied load, $K = 2.9/2 = 1.5K$

C = coefficient from Table 4 (p 4-80) for

$$R = 3 1/2 / 4 = 0.88$$

for $R = 0.88$, $\lambda = 0.25$

$$xL = (0.25)(4) = 1"$$

$$aL = 4" - xL = 3"$$

$$a = 0.75$$

$$C = 1.27$$

$$C_L = 0.86 \quad (p 4-74)$$

$$D = \frac{1.5}{(1.27)(0.86)(4)} = 0.777 \approx 1$$

$$D/t = 1 / (1/4) = 0.25 \quad OK - 0.25$$

6) $< 4 \times 4 \times 3/8 \times 4$ 2 plates, each $4 \times 3/8$ has $A = 3"$

$$F_u = 2.9/3 = 1.0K < 2.4 \quad OK \quad 0.04$$

7) attach weld, clip to beam -

Based on item 5, weld is OK, but check shear

$$A = 2 (1/4) [(2) (1/2) + 4] = 2.75$$

$$F_u = 2.9/2.75 = 1.05 < 2.4K < 2.4K \quad OK \quad 0.04$$

Node 235

3) attach weld, beam attach to beam

from ITT br. P471 for #16, 7/8 beam attach,
the welds are $2\frac{1}{2}"$.

$$A = (2\frac{1}{2}) (2) (14) = 1.25"$$

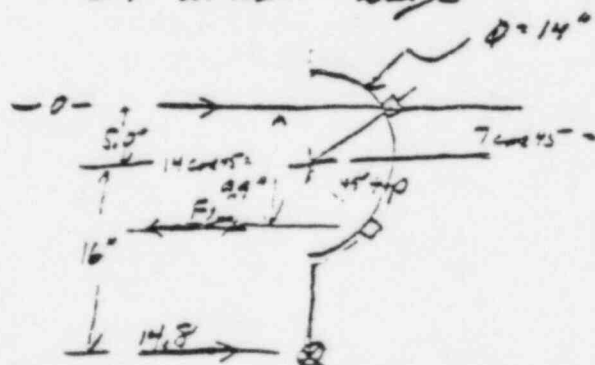
$$F_u = 2.9 / 1.25 = 232 < 27.9 \quad \text{OK } 0.10$$

Node 260/OP 157/ II-02-6005-4006 / 15725

There are elements to be analyzed:

- 1) Attachment lugs
- 2) Pipe clamps - Rayon-Patterson (assumed) 16" off by
1 1/2 x 5" stock
- 3) P.A. No. 2200-75 street
- 4) attach. weld, roof end of plate (this endures
all loadings in the street)

1) attach. lugs

 $\Sigma M_o:$

$$F_{lug} = \frac{(16-5) 14.8}{9.9} = 36.4$$

$$\text{from the lug code: } F/F_u = 0.52 < 0.5$$

2) Pipe clamps. Based on dimensions, this is a Rayon-Patterson
part No. 2370. (catalog dated 12/20/74)

$$F = 37.6 \text{ KSI}$$

$$F = 5.8 < 77.6 \quad \text{OK } 0.15$$

Node 260

3) B.P. - No. 220030 strut

from strut calc, $OK = 0.49$ 4) from the B.P. catalog, the welds are $7\frac{3}{8}"$ Long

$$A = \frac{1}{4} (7\frac{3}{8}) = 1.906 \text{ in}^2$$

$$F_y = 14.8 / 1.906 = 7.8 \text{ ksi} < 23.9 \text{ OK} = 0.33$$

Node 275 / dp 165 / II-02-6005-H003 / vertical guide

There are 4 elements to be analyzed:

- 1) The TS $4 \times 4 \times \frac{3}{4} \times 31.5"$ horizontal member
- 2) attachment weld of horizontal to vertical members
- 3) W6 $\times 25 \times 1\frac{1}{4}"$ vertical members
- 4) attach. weld of vertical member to structural steel.

The downward, shear design is acceptable by inspection -
the load is transferred directly to structural steel.

1) TS $4 \times 4 \times \frac{3}{4} \times 31.5"$ Long

			$L = 31.5"$
			$\alpha = 12^\circ$
			$F = 7.2$
			$S = 5.35$ (AISC 88, p 1-91)
A=?	31.500	RUN	
F=?	12.000	RUN	
S=?	3.200	RUN	
	5.350	RUN	

$$F_y = 2.8 < 40.6 \text{ OK } 0.07$$

$$F_y = A = 5.09$$

$$F_y = 7.2 / 5.09 = 0.6 \sim 0$$

R1 = 2.161
R2 = 1.839
S1 = -2.751
S2 = 2.446

Node 275

- 2) attachment weld
- $\frac{1}{4}$
- " fillet all around

Based on T3 $4 \times 4 \times \frac{1}{4}$ section, $S = 4.11 \text{ in}^3$
($A = 26.8 \text{ in}^2$)

$$F = 2.8 \frac{(5775)}{(4.11)} = 3.65 < 27.9 \quad \text{OK } 0.13$$

- 3) W6 x 25 vertical member

From AISC, $S^{xx} = 16.7$, $A = 7.34 \text{ in}^2$, $S = 16.7 \text{ in}^3$

$$F_a = 3.2 / 7.3 \sim 0$$

$$F_b = (2.8) \frac{(5775)}{(16.7)} = 0.9 \sim 0 \quad \text{OK}$$

- 4) attach weld, vertical member to civil steel -
-
- $\frac{1}{4}$
- " fillet on flanges & web.

Consider the flanges only.

$$b_f = 6.03 \text{ in}$$

$$t_f = 0.455 \text{ in}$$

$$d = 6.38 \text{ in}$$

$$I = 2 \left[\frac{1}{2} (b_f - t_f) \left(\frac{d}{2} \right)^2 \right]$$

$$I = 48.6$$

$$F_b = (2.8) (5775) \left(\frac{1}{48.6} \right) = 1.0 < 27.9 \text{ ksi} \quad \text{OK } 0.04$$

Node 285 / DP 175 / II-02-6005-H005 / strut

There are 2. elements to be analyzed

- 1.) Strut, Fig 211, page #3, CC-3'6 1/8"
- 2.) attach. weld to civil steel.

1) Sway Strut

From the sway strut calc., OK 0.47

2. Attach. weld: from the ITT Gr. PH 71 catalog,
the weld is 3" long.

$$A = 2(3)(3/8) = 2.25 \text{ in}^2$$

$$F = 5.5 / 2.25 = 2.5 \text{ ksi} < 27.9$$

OK 0.10

Node 290 / DP 177 / II-02-6005-H001 / Lat. guide

There are 4 elements to be analyzed

- 1) The W6x25 vertical member
- 2) the attach weld to the heavy member
- 3) The W6x25 horizontal member
- 4) The attach to civil steel.

1) W6x25 vertical member

$$M = (1444 + 178) 19.2 = 1464 \text{ K-in}$$

$$\text{from AISI, 2nd, } A = 7.34, S = 16.7$$

$$F_b = \frac{1464}{16.7} = 8.8 < 40.6 \text{ ksi} \quad \text{OK} \cdot 0.23$$

$$F_t = 19.2 / 7.34 = 2.6 < 27.4 \quad \text{OK} \cdot 0.11$$

Node 240

2) The connection weld

The weld is $\pm = 5/16$, all around, for eq. thickness of $5/8$ - this compares to $\pm = 5/16$, $\pm = 7/16$. Since the weld has a much larger section, use same stress

$$F = 8.8 < 23.9 \quad \underline{OK \quad 0.37}$$

3) The $W6 \times 25$ beam, member.

Treat as a continuous member; evaluate the intermediate weld only. This beam member will see the max moment applied by the next member so OK for vert. is OK for horiz. member. Also check for web crippling

Assume only flange welds contribute

$$d = 6.33$$

$$b_f = 6.08$$

$$I = 2 \left[\frac{1}{2} (6.33 - 0.25) \right]^2 (6.08)(0.25)$$

$$I = 28.6$$

$$F = \frac{(8.8)(167)(6.33/2)}{28.6} = 16.4 < 23.9 \quad \underline{OK \quad 0.67}$$

Web Crippling ($A = 16$, $r = 1.7$, $p = 5-47$)

$$A_{10} = \frac{P_{0F} - F_{yc} \pm (\pm_0 + 5R)}{F_{ys}}$$

$$F_{yc} = \text{cal } F_y = 36$$

$$F_{ys} = \text{act } F_y = 36$$

$$R = 17/16 \quad (p1-29)$$

$$P_0 = \text{flange force}$$

$$P_{0F} = 4/3 (8.8/6.33) = 1.9$$

$$\pm = 5/16$$

$$\pm_0 = 7/16$$

$$A_{10} = \frac{1.9 - 36(17/16)(7/16 - 5(17/16))}{36}$$

$$A_{10} = -1.75 < 0 \quad \text{no need for stiffener}$$

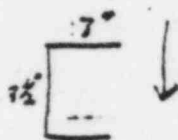
Node 290

4) The attachment to civil steel

load: from Rank, 5th, p103,

The max reaction force for an SS beam @
intermediate moment is $M_0/2$

$$R = 146.4 / (2'4" + 4" + 1'2") = 2.9 K$$

use table VIII of AISC, 9th (p 4-82), for clips wld.
on heavy member.

$$P = 2.9 K = 1.57 K$$

$$D = \frac{P}{C C_1 L} = 16 \text{ } \pm \text{ needed}$$

$$P = \text{applied load } (1/2 \text{ total} - 2 \text{ clips})$$

$$C_1 = 0.86 \text{ (p 4-78)}$$

$$a = 1.0 \text{ (conservative)}$$

$$R = 3/4 \text{ } \pm = 0.833$$

$$C = 0.833$$

$$L = 3.5$$

$$D = \frac{1.5}{(0.833)(0.86)(3.5)} = 0.56$$

$$t = 5/16, \quad \underline{OK} \quad 0.56/5 = \underline{0.11}$$

for clips wld on civil steel,

$$A = 2(5/16)(7) = 1.125$$

$$\bar{F} = 2.9 / 1.125 = 1.1 < 27.9 \quad \underline{OK} \quad \underline{0.05}$$

for the clips

$$A = 2(1/2)(7) = 7$$

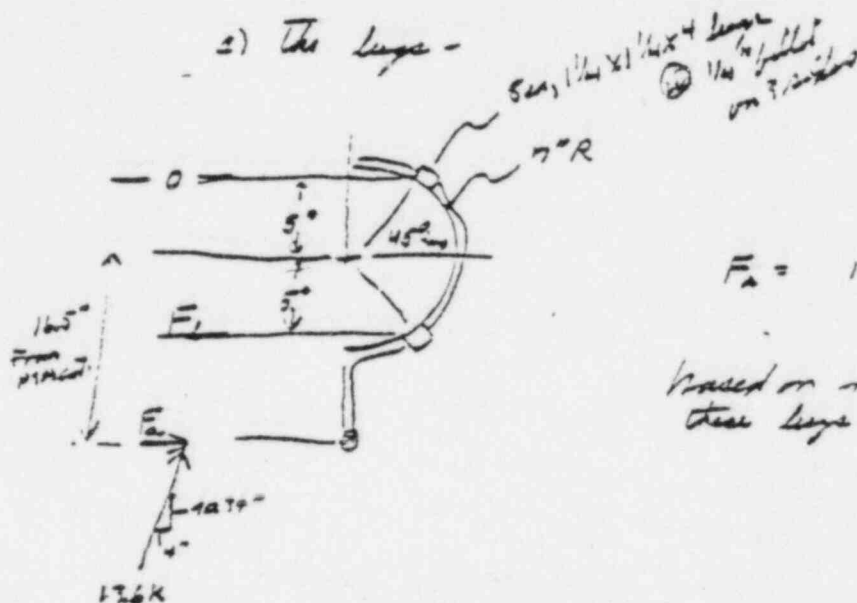
$$\bar{F} = 2.9 / 7 = 1.0 < 40.6 \quad \underline{OK} \quad \underline{0.03}$$

Node 310 / dp 146 / II-02-6005-H005 / snubber

There are elements to be analyzed

- 1) The lugs
- 2) The pipe clamp
- 3) The snubber
- 4) the clevis attach to wall stud

2) The lugs -



$$F_2 = \frac{17.6(4)}{40.74} = 0.6 K$$

Based on review of the lug calc,
these lugs are negligibly loaded.

OK - 0.0

2) The pipe clamp

from 4/20/74 PIA catalog,

for S = arstan (4/20.74) = 2.5

$$F = 45.4 K @ 5^\circ$$

$$\bar{F} = 17.6 < 45.4 \quad \text{OK} \quad \underline{0.30}$$

3) The snubber

same catalog, $F = 72.4 K$

$$F = 17.6 < 72.4 \quad \text{OK} \quad \underline{0.10}$$

Node 310

4) The attach to civil steel

Of the box section is assumed to be built up of
full per. welded $1/2"$ plates

$$A = 4(7)(1/2) = 14 \text{ in}^2$$

$$F = 126/14 = 9.07 \text{ ksi} < 23.9 \quad \text{OK } 0.04$$

Node 315 / dp 195 / II-02-6005-H-002 / strut

There are 6 elements to be analyzed:

- 1) the strut
- 2) attach weld to W6x15 horizontal member
- 3) W6x15 horiz. member
- 4) attach weld to vert. member
- 5) vert. W4x13 member
- 6) attach weld to civil steel.

1) the strut.

Per the swing strut calculation. OK - 0.80

2) attach weld to W6x15 horizontal member

 $1/4"$ all around on $7-1/2 \times 3$ plate

$$A = 2(1/4)(4-1/2 + 3) = 3.75 \text{ in}^2$$

$$F = 9.4/3.75 = 2.5 \text{ ksi} < 23.9 \quad \text{OK } 0.11$$

3) W6x15 horiz. member

	23.888	RUN
Q=?		
	14.888	RUN
F=?		
	9.488	RUN
S=?		
	9.720	RUN

R1 = 4.788
R2 = 4.788
S1 = -2.388
.. = ..

$$L = 28'$$

$$a = 14$$

$$F = 9.4$$

$$\phi = 9.72 \quad (\text{AISC, 8th, p1-28})$$

$$F_b = 3.4 < 40.6 \quad \text{OK } 0.08$$

$$A = 4.43$$

$$F_v = 9.4/4.43 = 2.1 < 24.4 \quad \text{OK } 0.09$$

Node 315

4) Attach. weld to W4X17 member

for W4X17, $t_f = 1/4$, $t_w = 3/4$. Since the flange
makes predominant contribution to I , use W4X13
 I for weld section. $I = 1.90$

$$\bar{F}_b = \frac{(24)(4.72)}{(1.90)} = 17.4 < 27.9 \text{ OK } 0.73$$

for W4X17, $d = 7.16$, $b_f = 4.06$

$$A = 1/4 (2(4.06) + 7.16) = 3.07 \text{ in}^2$$

$$\bar{F}_a = 4.7 / 3.07 = 1.53 \text{ ksi} < 23.9 \text{ OK } 0.66$$

$$\text{interaction } 0.73 + 0.66 = 0.79 < 1 \text{ OK}$$

5) W4X13 member

same stress as calc'd for 4) above:

$$\bar{F}_b = 17.4 < 40.6 \text{ OK } 0.43$$

$$A = 3.83 \text{ in}^2$$

$$\bar{F}_a = 4.7 / 3.83 = 1.2 < 40.6 \text{ OK } 0.07$$

$$\text{interaction} = 0.46 \text{ OK}$$

6) attach weld to end stud

this weld is enveloped by item 4). The only difference
is that the web has a $5/16"$ weld vs $1/4"$ for item 4

$$\text{OK } 0.79$$

Node 330 / d.p 205 / II-06-6005-H008 / first

This support has 4 elements to analyze

- 1) Clamps
- 2) strut
- 3) clavis
- 4) attach weld to civil steel

1) Clamps -

for $1\frac{1}{4} \times 5$ inch stock, 3.P (see photo) strut hooked to
3" od strut, @ adjustment, know a
BP 2200-30 strut -

for $1\frac{1}{4} \times 5$ inch stock, 3 bolt, know BP # 2600
For 14" - $F = 326 K$

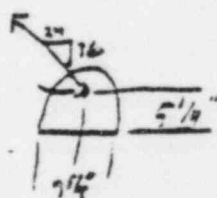
$$F = 133 < 326 \quad \text{OK } 0.75$$

2) strut & clavis

B.P. 2200-30 reg. includes strut & clavis:

Per the strut table OK 0.44

4) attach. weld to civil steel



$7\frac{1}{4} \times 5 \times \frac{5}{16}$ weld

$$A = 2(7\frac{1}{4})(7\frac{1}{4} + 5) = 7.89 \text{ in}^2$$

$$F_a = 133(\cos 37^\circ) / 7.89 = 1.40 \text{ K.I}$$

$$I = 2 \left[\frac{1}{2} (7\frac{1}{4}) \right]^2 (5) (\frac{5}{16}) = 45.4$$

$$F_b = \frac{133(\sin 37^\circ) (\frac{1}{2} (7\frac{1}{4}))}{45.4} = 0.6$$

$$F = F_a - F_b = 2.0 \text{ K.I} < 27.9 \quad \text{OK } 0.08$$

F_y negligible OK

Nide 340 / dp 210 / — snubber

This suspect has elements to be analyzed:

- 1) large
- 2) clump -
- 3) snubber away
- 4)

can't find matching clump
clump trace w/ to missing

Node 350

2) Scrubber assay -

Per the scrubber table, OK - 1.11

Since the scrubber was oriented off the $\perp$ to the pipe & load rating on the clamp needs checked:

$$\theta = \sin^{-1} \left[\frac{2'4''}{7'0\frac{1}{4}''} \right] = 50.8^\circ$$

per the Pres. Sec. Data:

S	F
45	7435
50.8	6907 =
60	6070

$$\text{for faulted } F = 1.55 (6907) = 10706$$

$$F/F = 19/10.7 = 1.78 \quad \underline{NG}$$

Node 405 / dp 250 / SI-06-6005-H003 / vert guide

This support has 6 features to be analyzed:

- 1) W6X15X2'3" top horiz member
- 2) attach weld to vertical members
- 3) W6X15X2'4 vert members
- 4) attach weld to baseplate
- 5) baseplate
- 6) anchor bolts.

the bottom horiz. member will be analyzed by analysis of top horiz. member

1) W6X15X2'3" top horiz member

for W6X15:

$$\begin{aligned} D &= 6.28 \\ b_p &= 4.03 \\ t_w &= 1/4 \\ t_f &= 3/8 \\ A &= 7.74 \\ S &= 10.2 \\ R &= 3/4 \end{aligned}$$

$$\begin{aligned} L &= 2'3" + 6.28" = 37.28" \\ A &= 1/2 L = 16.64 \\ F &= 29.4 \\ S &= 10.2 \end{aligned}$$

$$F_b = 12.0 < 42.6 \text{ ksi OK } 0.30$$

$$F_y = 14.7 / 7.74 = 3.1 < 24.4 \text{ ksi OK } 0.13$$

check for web crippling:

$$17.5 \sim \frac{D_p F - F_{yc} t}{F_{yt}}$$

$$F_{yc} = F_{yt} = 36 \text{ ksi}$$

$$R = 0.75$$

$$D_p F = (4/3) 29.4 = 39.2$$

$$t = 1/4$$

$$t_b = 1/16 \text{ assumed contact zone}$$

Node 405

$$F_{yt} = \frac{79.2 - 36(1/16)(1/16 + 5(7/16))}{36}$$

$$A_{yt} = 0.136 \text{ in}^2$$

the 2 $3/8 \times 2 1/2 \times 5 1/2$ " plates have

$$A = (2)(7/8)(2 1/2) = 1.875 > 0.136 \quad \underline{\text{OK 0.07}}$$

2 attach weld to vertical members.

the weld throat, $1/4$ "
 joint $t_p = 3/4$ $\Rightarrow$ $1/4$ allows using the cable's beam
 stress for the weld

$$F_b = 120 < 27.9 \quad \underline{\text{OK 0.50}}$$

$$F_y = 3.1 < 27.9 \quad \underline{\text{OK 0.13}}$$

3) vertical member -

same section as the horizontal member, same
 moment:

$$F_b = 12 < 40.6 \quad \underline{\text{OK 0.3}}$$

$$F_a = F_y(2) = 3.1 < 40.6 \quad \text{OK 0.08}$$

$$\text{interaction: } 0.30 + 0.08 = \underline{0.38 \text{ OK}}$$

4) attach weld to baseplate

same as 3 above, but applied to (2)

$$F_b = 120 < 27.9 \quad \text{OK 0.50}$$

$$F_a = F_y(2) = 3.1 < 27.9 \quad \text{OK 0.13}$$

$$\text{interaction: } 0.50 + 0.13 = \underline{0.63 \text{ OK}}$$

Node 405

5) Baseplate

treat as a beam, and use symmetry.



$$M_0 = 14.7(6) = 88.2 \text{ K-in}$$

$$S = \frac{1}{12} \left[\frac{1}{2} (14)(11)^3 \right] = 2.33$$

$$F_b = 88.2 / 2.33 = 37.9$$

$$F_b = (0.83)(0.75)(36) = 50.76 \quad \text{OK } 0.75$$

$$F_b = 14.7 / 12 = 1.3 < 27.4 = \text{OK } 0.05$$

6) Anchor bolts:

$$1\frac{1}{4} \text{ CEB, min embed } 6" - F = 19K$$

$$F = 14.7 / 2 = 7.4 < 19 \quad \text{OK } 0.75$$

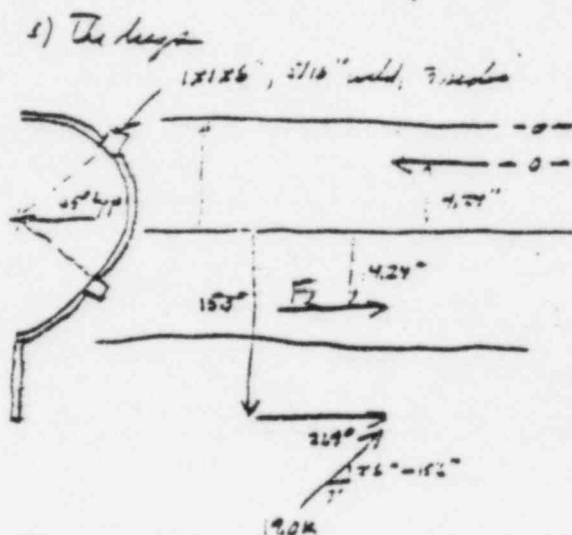
Node 445 / dp 790 / SI-06-0719-4004 / SPR HSR

Per the spring hanger table - OK - 0.78

Node 455 / dp 577 / SI-06-0719-4007 / snubber

There are features to be analyzed:

- 1) The legs
- 2) The change
- 3) The snubber Assy
- 4) clavis attach weld
- 5) the plate
- 6) The snubber rollers



$$F_2 = (19.0K \cos 26.4^\circ) \frac{(153^\circ - 4.29^\circ)}{(26.4^\circ)}$$

$$F_2 = 41.0K$$

From the leg calc OK 0.57

- 2) The change

for a P.C. - 24-120, @ Load @ 68.1°, $F = 1.55(17404) = 26.5K$

$$F = 19.0 < 26.5 \quad \text{OK } 0.88$$

- 3) The snubber Assy.

from the snubber table, OK - 0.26

Node 485

4) Clavis attach weld

The weld pattern, based on an earlier JTB bracket, is a 7" x 7" x 5/16" weld pattern.

$$A = 5/16 (7+7)(2) = 8.75 \text{ in}^2$$

$$I = 2 \left(\frac{1}{2}(7) \right)^2 (7)(5/16) = 88.6$$

$$S = 88.6 / (2)(7) = 19.7$$

$$F_a = (19)(\cos 21.9) / 8.75 = 2.0 \text{ ksi}$$

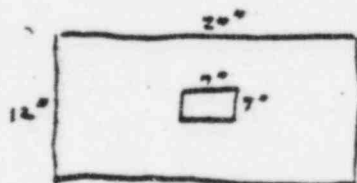
$$F_b = (19)(\sin 21.9)(4.3) / 19.7 = 1.73$$

$$F_y = (19)(\sin 21.9) / 8.75 = 0.810$$

$$F = [(2.0^2 + 1.73^2) + 0.81^2]^{1/2} = 3.8 < 23.9 \quad \text{OK 0.16}$$

5) the plate

pull-out load - assume SS - uniform central pressure



From Rank, 5th, p 786, fig 1c

$$a = 24$$

$$a = b$$

$$b = 12$$

$$a/b = 0.58 - 0.6$$

$$a/b = 0.58 - 0.6$$

$$b/b = 0.58 - 0.6$$

$$b/b = 0.58 - 0.6$$

$$a/b$$

$$T_{max} = T_0 = \frac{P_w}{S} = \frac{(0.6)(174 \sin 21.9)}{(74)^2}$$

$$T_0 = 18.8 \text{ ksi}$$

bending load: assume supported by outer holes only:



$$R = [19(\sin 21.9)(4.3)] / 24$$

$$R = 0.3 \text{ K}$$

$$M = (0.3)(12) = 3.5 \text{ in-k}$$

$$S = \frac{1}{(4)(24)} (12)(7/4)^3 = 13.5$$

$$F_b = 3.5 / 13.5 = 0.3$$

$$F_b = 18.8 + 0.3 = 19.1 < 40.6 \quad \text{OK 0.47}$$

Npdcwss

Anchor bolts: for 1 1/4" C.B. @ 5 3/4" embedment in 2400
 concrete $F_u = 1/4(19.0) = 4.8K$
 $F_t = 1/4(76.75) = 9.2K$

assume central bolts carry pull-out

$$F = 19.0/2 = 9.5 > 4.8 \quad \underline{NG 1.98}$$

need a detailed plate analysis, all bolts will be
 in tension, so $\phi = 2$ is NG.

Node 440 / dp 537 / II-05-079-H506 / snubber

This support has features to be analyzed:

- 1) lugs
- 2) clamps
- 3) snubber Assy.
- 4) clevis attach
- 5) baseplate
- 6) anchor bolts.

1) lugs

assume full axial load on 1x1x6 lugs

$$F = (17.7K) \left[\frac{5'4''}{6'6''} \right] = 14.5K$$

per the lug code OK - 0.72

2. clamps

the clamp is a PSA-35 loaded at an angle of

$$\theta = \sin^{-1} \left[\frac{5'4''}{6'6''} \right] = 55^\circ$$

from the PSA catalog, $F = (1.55)(14.72) = 22.8K$

$$F/F = 17.7/22.8 = \underline{0.78}$$

3. snubber Assy

per the snubber table OK - 0.24

4, 5, 6 - Base attach - clevis attach, baseplate, anchor bolts.

Per FCR-X-41-747-S stress ratios
with a load of 17.0K is 0.87.

$$\therefore \text{OK: } 0.87 \left(\frac{17.7}{19.0} \right) = \underline{0.81}$$

Node 520/dp 540 / SI-06-0327-H013 / 1st hanger

The piping was designed to function successfully w/o this support - analysis not needed.

Node 675/dp 775 / SI-06-0719-H005 / 1st hgr

From the spring hanger Table, OK 0.86

Node 710 / dp 455 / SI-02-6008-H001 / guide

This support is identical to support at node 720 (dp 460), except the loads differ:

	Loads Node 710	Loads Node 720
Horizontal	1.7	1.6
Vertical	2.5	1.4

The maximum increase in load gives a factor of 1.1

The maximum stress ratio @ node 720 is 0.06

Therefore $U = 1.1 (0.06) = 0.06$

Note 720 / dp 460 / SE-02-6003-H002 / guide

There are 6 elements to be analyzed:

- 1) The horizontal $W4 \times 13 \times 1'2\frac{3}{4}"$ beam restraining uplift
- 2) attachment weld, item 1 to the vertical $W4 \times 17 \times 3'4"$ sides of the box
- 3) The vertical $W4 \times 17 \times 0'5"$ beams transferring horizontal loads to item (1)
- 4) the vertical $W4 \times 17 \times 2'4"$ beams comprising the sides of the box
- 5) attachment weld, item 4 to item 6
- 6) the 4" sch 40 pipes resisting down loading.
(welds hold in place & do not see the load)

1) This $W4 \times 13 \times 1'2\frac{3}{4}"$ beam will be modeled Fixed-fixed with a central uplift load.

$L = ?$			$L = 1'2\frac{3}{4}" = 19.75"$
$a = ?$	14.750	RUN	$a = \frac{1}{2}L = 9.875"$
$F = ?$	7.375	RUN	$F = 1.4$
$S = ?$	1.488	RUN	$S = 5.46 \text{ in}^2, A = 7.83 \text{ in}^2$ (AISC, 5 th , p 1-25)
	5.468	RUN	$F_u = 0.7 / 7.83 = 0.089 \text{ ksi} < 24.4 \text{ ksi} \text{ OK}$
$P1 = 0.788$			$F_b = 0.5 \text{ ksi} < 40.6 \text{ ksi} \text{ OK 2.01}$
$P2 = 0.788$			
$S1 = -0.473$			
$SA = 0.473$			
$SL = -0.473$			

Node 720

2) Attach weld, element (2) to element (3)

The weld is a $1/4"$ fillet on web & both flanges. This compares to a beam section with $t_w = 1/4$, $t_f = 3/8"$. For bending, can assume flange controls section properties: use beam stress

$$F_b = 0.51 KSI < 23.9 \quad OK \quad 0.02$$

3) Vertical W4X13 beams used as compression members.

This element will be analyzed using 1.15.5.2 of AISC, 8th, p 5-47. This section makes provisions for local web buckling due to local compression loading from an attached beam in bending.

$$F_{yc} = F_{yt} = 36 KSI$$

$$R = 17/16"$$

$$P_{bf} = 4/3 (1.6) = 2.1 KSI$$

$$t_b = 0.23$$

$$t_b = 1/12" \text{ (estimated width of line of contact between pipe and beam)}$$

$$A_{st} = \frac{P_{bf} - F_{yc} t_b (t_b + 5R)}{F_{yt}}$$

$$A_{st} = \frac{(2.1) - (36) (0.23) (1/12 + 5(17/16))}{36}$$

$$A_{st} = -1.1 in^2$$

since $A_{st} < 0$, no web stiffener is needed.

4) vertical W4X13X2'4" box sides.

L=?

28.888 RUN

R=?

18.688 RUN

F=?

1.688 RUN

S=?

5.468 RUN

Analyze as a F-F beam, $F_a = 0.3 KSI$ (see element 12) with a transverse load of 1.8 K applied $(31' - 5' 1/2") - (29' - 10' 3/4") = 18.56"$ above the base.

$$L = 2'4" = 28$$

$$a = 18.6"$$

$$F = 1.6 K$$

$$S = 5.46"$$

$\sigma_1 = 8.428$
 $\sigma_2 = 1.188$
 $\sigma_3 = -9.614$
 $\sigma_4 = 9.316$
 $\sigma_5 = -1.216$

Node 720

$$F_0 = 618 / 7.83 = 0.3 \text{ ksi} < 24.4$$

All stresses are very low - OK. 0.01

5) attachment weld, item 4 to item 6.

Same as item 2, use beam stress:

$$F_0 = 1.7 < 23.9 \quad \text{OK} \quad 0.05$$

6) 4" sch 40 pipe

The pipe is too short to buckle

$$F_a = 1.43 / 7.17 = 0.4 \sim 0.0 \quad \text{OK}$$

Node 755 / DP 425 / SI-02-6006-H001 / guide
 765 430 -6006-H002
 800 405 -6007-H001
 810 710 -6007-H002

These 4 supports are identical to SI-02-6008-H002.
 They will be evaluated by comparing loads, as was done for
 node 710

Node	load		Factor	F/F	
	Horizontal	Vertical			
720	1.6	1.4	1.0	0.05	
755	1.5	4.7	3.4	0.17	OK
765	1.8	0.9	1.2	0.06	OK
800	1.8	5.8	4.2	0.21	OK
810	1.6	1.2	1.0	0.05	OK

Node 846 / dp - / - / 2 rubber

This rubber, inclined to ground on the east end
 FW heavy piping, has no slugs available.

Node 855 / dp 710 / SI-05-0720-H017 / spring hanger

Per the spring hanger table - OK 0.87

Nide 755 / dp 312 / -05-0320-4016 f5 naber.

This support has features to be evaluated:

- 1) legs
- 2) clamp
- 3) snubber Assy
- 4) attach weld for clavis
- 5) plate
- 6) anchor bottom

1) legs -

legs are on snubber $\pm$;
 Per. sec clamp has 15.5" $\pm$ offset

$$F_{leg} = [7.2 \sin(47.2)] \left[\frac{6 - 15.5}{12} \right] = 9.5 K$$

per the leg calculation - OK 0.28

2) clamp

the std. PSC clamp for the application @ 45°
 has $F = 1.55 (18) = 27.9$

$$F = 7.3 < 27.9 \quad \text{OK } 0.26$$

3) snubber Assy.

per the snubber table - OK 0.10

4) the plate -

assume a 33 plate @ central pt load:

$$F = 7.2 \sin 47^\circ \approx 5.3 K$$

dimensions 22" x 20"
 7" x 7" contact pressure

Node 555using Rank, 5th, fig 10 p 386:

$$W = 5.3$$

$$a/b = 7/20 = 0.35 \sim 0.4$$

$$b/b = 7/20 = 0.35 \sim 0.4$$

$$a/b = 22/20 = 1.10 \sim 1.0$$

$$\beta = 0.84$$

$$\bar{F}_b = \frac{\beta W}{\beta^2} = \frac{(0.84)(5.3)}{(0.84)^2} = 7.9 \text{ ksi}$$

$$\bar{F}_b = 7.9 < 40.6 \quad \underline{\text{OK}} \quad \underline{0.20}$$

5) The anchor bolts -

for 1" CCB, 4 1/2" anchor.

$$\bar{F}_t = 14,200/4 = 3.5 \text{ K}$$

$$\bar{F}_v = 27.4/4 = 6.8 \text{ K}$$

$$\text{pull out load: } \bar{F}_t = \frac{1}{4}(5.3) = 1.3 \text{ K}$$

bending: assume bending moment taken by opposite bolts:

$$\bar{F} = \frac{(7.2 \cos 47.2)(4.8)}{((22+20)^{1/2})} = 0.8 \text{ K}$$

$$\text{total: } \bar{F}_t = 6.3 + 0.8 = 7.1 < 7.5 \quad \underline{\text{OK}} \quad \underline{0.60}$$

$$\text{shear: } \bar{F}_v = \frac{1}{4}(7.2 \sin 47.2) = 1.32 < 6.8 \text{ OK } 0.19$$

$$\text{interaction: } (0.6^2 + 0.19^2)^{1/2} = 0.63 < 1 \quad \underline{\text{OK}} \quad \underline{0.63}$$

Node 361 / dp 315 / II - 05 - 0720 - H011 / snubber

Three elements need analysis:

- 1) the clamp
- 2) the snubber
- 3) the attachment structure (clevis, frame, bracket)

1) The clamp -

use a 12" Ø PIA-35 clamp w

$$S = 31.5" \left[\frac{20' 7\frac{1}{2}" - 16' 6"}{5' 7"} \right] = 42.8"$$

from PIA catalog:

$$F = (1.55)(18.0) = 27.9$$

$$F/F = 6.2/27.9 = 0.22 \text{ OK}$$

2) The snubber -

per the snubber table, OK = 0.09

3) The attachment structure

Review of F.R. X-41-763-S showed it applicable, except that an F0302 was used on the anchor bolt. Correction for this gives $F/F = (0.747)(2) = 1.49$ with a 14 K snubber load.

$$F/F = (1.49) \left(\frac{6.2}{14.2} \right) = 0.66$$

Node 870 / Sp 320 / E-05-0720-4009 / snubber

There are 3 elements to be evaluated:

- 1) clamps
- 2) snubber arms
- 3) clevis attach weld

1) The clamp - the std -120 PSC clamp for the -35 snubber, @ $\theta = 40^\circ$, has $F = (1.55) 50 = 77.5$

$$F = 12.5 < 77.5 \quad \text{OK } 0.16$$

2) The snubber

Per the snubber table $\text{OK } 0.17$

3) The clevis attach weld:

for a $7 \times 7 \times 5/16$ weld,

$$A = 5/16 (7+7) 2 = 8.75"$$

$$S = \frac{1}{\frac{1}{2}(7)} \left(\frac{1}{2}(7) \right)^2 (7(5/16)) = 9.84$$

The snubber is offset from $\perp$ by atan $\left(\frac{4.375}{7.41} \right) = 37.4^\circ$

$$F_a = 12.6 (\cos 37.4) / 8.75 = 1.2 < 40.6 \quad \text{OK } 0.03$$

$$F_o = 12.6 (\sin 37.4) (4.8) / 9.84 = 3.4 < 40.6 \quad \text{OK } 0.08$$

$$\text{interaction: } 0.08 + 0.03 = \text{OK } 0.11$$

SI-51 Support Loads (ABUR, 10/5/55; ADOG, 10/16/55)

node	dcr	wt	T	$(\text{sum of } \Delta H)^2$	'Level A, 2'	'Level O'
22	Y	—	—	9193	—	± 9193
100	I	—	—	5028	—	± 5028
115	X	173	1176	5439	(133, 1304)	(-5706, 6748)
115	Y	3711	-431	7843	(2330, 7311)	(-5463, 11154)
120	Y	1784	182	4161	(1784, 1526)	(-2777, 5727)
125	X	247	-1426	4910	(-1189, 247)	(-6049, 5757)
175	I	-312	-179	12215	(-712, -133)	(-1227, 12082)
145	Y	2482	263	7499	2745	(-5017, 10244)
170	X	—	—	12673	—	± 12673
180	Y	2393	-175	7663	2393	(-4445, 10556)
195	I	-483	1524	8125	(-483, 1841)	(-8658, 4216)
200	Y	4143	-317	7717	4143	(-3591, 11860)
235	Y	1223	764	4238	1587	(-7015, 5825)
260	I	-200	-1760	12691	-2060	(-14757, 12491)
275	Y	3648	249	6854	3897	(-3706, 10751)
285	I	540	-252	4960	540	(-4672, 5500)
290	I	28	1565	17623	1593	(-17595, 19216)
310	I	—	—	13605	—	± 13605
315	Y	2890	287	6167	3177	(-7277, 4344)
330	I	-232	-109	12916	-391	(-17307, 12634)
340	I	—	—	16713	—	± 16713
345	Y	3774	-2132	7712	3774	(-9470, 17046)
375	I	—	—	40189	—	± 40189
380	I	—	—	18987	—	± 18987
405	Y	4141	689	24580	4970	(-20419, 24790)
485	I	—	—	18970	—	± 18970
490	I	—	—	17653	—	± 17653
520	Y	5550	-404	16504	5550	(-11358, 22054)
575	I	—	—	10576	—	± 10576
610	Y	4155	-471	8910	4155	(-9226, 7065)
625	I	—	—	17552	—	± 17552
710	Y	481	-72	1920	481	(-1571, 2461)
710	I	134	51	1525	135	(-1391, 73)

II-51 Support Loads (cont'd)

node	dir	WT	T	(1982/8/23) 1/2	'Level A3'	'Level D'
720	Y	255	-59	1165	255	(-969, 1425)
720	Z	-36	7	1536	-36	-1572
755	Y	1419	142	3059	1611	(-1618, 4670)
755	Z	85	336	1119	421	(-1034, 1540)
765	Y	143	-8	724	143	(-649, 427)
765	Z	-28	-44	1704	-72	(-1776, 1676)
800	Y	1593	-150	4228	1593	(-2735, 5321)
800	Z	84	603	1095	687	(-1011, 1792)
810	Y	90	-103	1137	(-13, 90)	(-1150, 1227)
810	Z	-18	-107	1599	-125	(-1727, 1581)
840	H	—	—	6362	—	= 6362
855	H	—	—	7251	—	= 7251
861	H	—	—	6206	—	= 6206
870	H	—	—	12533	—	= 12533
886	X	313	670	16572	983	(16.99, 17495)
886	Y	1398	-1156	3352	1398	(-310, 4750)

SONGS 1 SI-51 ANALYSIS IMPELL SAHS

NODE	TYPE	REACTIONS (LBS OR IN-LBS)	DIRECTION
1	MOMENT	697796.	Z COORD
2	FORCE	108340.	Y COORD
3	FORCE	30654.	X COORD
4	MOMENT	430673.	Z COORD
5	MOMENT	1036303.	Y COORD
6	MOMENT	226376.	X COORD
7	FORCE	11980.	Z COORD
8	FORCE	1525.	Y COORD
9	FORCE	1165.	X COORD
10	FORCE	1532.	Z COORD
11	FORCE	0683.	Y COORD
12	FORCE	582.	X COORD
13	FORCE	919.	Z COORD
14	FORCE	7212.	Y COORD
15	FORCE	74875.	X COORD
16	MOMENT	33169.	Z COORD
17	MOMENT	33059.	Y COORD
18	FORCE	1119.	X COORD
19	FORCE	1786.	Z COORD
20	FORCE	1704.	Y COORD
21	FORCE	2522.	X COORD
22	FORCE	2522.	Z COORD
23	FORCE	1077.	Y COORD
24	MOMENT	5683.	X COORD
25	MOMENT	45181.	Z COORD
26	MOMENT	27197.	Y COORD
27	FORCE	42220.	X COORD
28	FORCE	1095.	Z COORD
29	FORCE	1137.	Y COORD
30	FORCE	1599.	X COORD
31	FORCE	5980.	Z COORD
32	FORCE	453.	Y COORD
33	FORCE	1910.	X COORD
34	FORCE	3989.	Z COORD
35	MOMENT	80401.	Y COORD
36	MOMENT	25672.	X COORD
37	FORCE	6362.	INCLINED
38	FORCE	7251.	INCLINED
39	FORCE	6206.	INCLINED
40	FORCE	12533.	INCLINED
41	FORCE	16512.	INCLINED
42	FORCE	33352.	INCLINED
43	FORCE	3935.	INCLINED
44	FORCE	105.	INCLINED
45	FORCE	2562.	INCLINED
46	FORCE	2562.	INCLINED
47	FORCE	2562.	INCLINED
48	FORCE	2562.	INCLINED
49	FORCE	2562.	INCLINED
50	FORCE	2562.	INCLINED
51	FORCE	2562.	INCLINED
52	FORCE	2562.	INCLINED
53	FORCE	2562.	INCLINED
54	FORCE	2562.	INCLINED
55	FORCE	2562.	INCLINED
56	FORCE	2562.	INCLINED
57	FORCE	2562.	INCLINED
58	FORCE	2562.	INCLINED
59	FORCE	2562.	INCLINED
60	FORCE	2562.	INCLINED
61	FORCE	2562.	INCLINED
62	FORCE	2562.	INCLINED
63	FORCE	2562.	INCLINED
64	FORCE	2562.	INCLINED
65	FORCE	2562.	INCLINED
66	FORCE	2562.	INCLINED
67	FORCE	2562.	INCLINED
68	FORCE	2562.	INCLINED
69	FORCE	2562.	INCLINED
70	FORCE	2562.	INCLINED
71	FORCE	2562.	INCLINED
72	FORCE	2562.	INCLINED
73	FORCE	2562.	INCLINED
74	FORCE	2562.	INCLINED
75	FORCE	2562.	INCLINED
76	FORCE	2562.	INCLINED
77	FORCE	2562.	INCLINED
78	FORCE	2562.	INCLINED
79	FORCE	2562.	INCLINED
80	FORCE	2562.	INCLINED
81	FORCE	2562.	INCLINED
82	FORCE	2562.	INCLINED
83	FORCE	2562.	INCLINED
84	FORCE	2562.	INCLINED
85	FORCE	2562.	INCLINED
86	FORCE	2562.	INCLINED
87	FORCE	2562.	INCLINED
88	FORCE	2562.	INCLINED
89	FORCE	2562.	INCLINED
90	FORCE	2562.	INCLINED
91	FORCE	2562.	INCLINED
92	FORCE	2562.	INCLINED
93	FORCE	2562.	INCLINED
94	FORCE	2562.	INCLINED
95	FORCE	2562.	INCLINED
96	FORCE	2562.	INCLINED
97	FORCE	2562.	INCLINED
98	FORCE	2562.	INCLINED
99	FORCE	2562.	INCLINED
100	FORCE	2562.	INCLINED

SONGS 1 SY-51 ANALYSIS IMPELL SAHS

SUPPORT REACTIONS FOR LOAD COMBINATION CASE NO. 27

SSE + SAHS (SRSS IS W AND W/O ROD FINGERST)

NODE	TYPE	REACTIONS (LBS OR IN-LBS)	DIRECTION
21	FORCE	21270.	X COORD
21	FORCE	8876.	Y COORD
21	FORCE	15422.	Z COORD
21	MOMENT	691290.	X COORD
21	MOMENT	1022028.	Y COORD
21	MOMENT	277081.	Z COORD
22	FORCE	276193.	Y COORD
22	FORCE	5028.	INCLINED
22	FORCE	5439.	Y COORD
22	FORCE	7053.	Y COORD
22	FORCE	4161.	Y COORD
22	FORCE	4910.	X COORD
22	FORCE	12215.	INCLINED
22	FORCE	7569.	Y COORD
22	FORCE	12673.	Y COORD
22	FORCE	7663.	Y COORD
22	FORCE	6175.	Y COORD
22	FORCE	7777.	Y COORD
22	FORCE	4238.	Y COORD
22	FORCE	12691.	Z COORD
22	FORCE	6854.	Y COORD
22	FORCE	4960.	Z COORD
22	FORCE	12623.	Y COORD
22	FORCE	13605.	INCLINED
22	FORCE	6167.	Y COORD
22	FORCE	12016.	INCLINED
22	FORCE	16713.	INCLINED
22	FORCE	9712.	Y COORD
22	FORCE	40189.	INCLINED
22	FORCE	18567.	INCLINED
22	FORCE	45560.	Y COORD
22	FORCE	18970.	INCLINED
22	FORCE	17653.	INCLINED
22	FORCE	16505.	Y COORD
22	FORCE	10596.	INCLINED
22	FORCE	81.	X COORD
22	FORCE	4531.	Y COORD
22	FORCE	1779.	Z COORD
22	FORCE	8918.	Y COORD
22	FORCE	17552.	INCLINED
22	FORCE	16677.	X COORD
22	FORCE	5060.	Y COORD
22	FORCE	8122.	Z COORD
22	MOMENT	347296.	X COORD
22	MOMENT	177143.	Y COORD

7I-51 Support Loads ~~ABQR~~ ABUR

851219.

NAEPH-11L, NUCLEAR SERVICES CORPORATION AIRING ANALYSIS PROGRAM - VERSION 1.9.1

SONGS : 51-54 ANALYSIS IMPULS SAYS (IMPULS, HORIZ. SPECTRA)

SUPPORT REACTIONS FOR LOAD CASE NO. 1

WEIGHT PLUS EXTERNAL FORCES

[illegible]

4/2/86

HP 41C
code
for Load
combination

PRP *LCON*

Combines pipe support loads by:
Level A,B min = min (W, W+T)
max = max (W, W+T)

01 *LBL *LCON*

02 * WT?

03 PROMPT

04 STO 01

05 STO 02

06 * T?

07 PROMPT

08 ST+ 02

09 RCL 01

10 RCL 02

11 X<Y?

12 X<Y

13 STO 03

14 X<Y

15 STO 04

16 *A, MN=

17 ARCL 03

18 PROMPT

19 *A, MX=

20 ARCL 04

21 PROMPT

22 *SEIS?

23 PROMPT

24 ST- 03

25 ST+ 04

26 *B, MN=

27 ARCL 03

28 PROMPT

29 *B, MX=

30 ARCL 04

31 PROMPT

32 END

WT

Thermal

Total min = Level A,B min = 11E-107
max = Level A,B max = 11E-107

→ A, MN = Level A, B, min

→ A, MX = Level A, B, max

→ B, MN = Level B, min

→ B, MX = Level B, max

LUSS

The following data has been gathered from the support analyses:

Node / ID	Force	-ll direction length	both sides welded?	⊥ dir length	weld throat
100/75	121 K	(L ₂) 4"	✓	(L ₁) 1 1/4"	1/4"
175/45	23.7 K	4"	✓	1 1/4"	1/4"
170/110	13.7	2"	✓	2"	1/4"
	13.7	4"	✓	1 1/4"	1/4"
200/157	31.4	4	✓	1 1/4"	5/16
780/270	47.4	4	✓	1 1/4"	(use 1/4) 5/16
485/577	41.0	6	✓	1"	5/16
790/577	14.5	6	✓	1"	5/16
746/					
855/312	9.5	6	✓	1"	1/4"

LUGS

ASME Code Case N-918-2 shall be used, (4/12/84), but only
 sym 9 stresses will be checked

1.0 Limitations

1.1 fillet welds ok

1.2. must sit - cold piping

1.3 $\beta_1 \leq 0.5$, $\beta_2 \leq 0.5$, $\beta_1 \beta_2 \leq 0.075$

$$\beta_1 = L_1 / r, \quad \beta_2 = L_2 / r$$

L_1 = width of plate r = mean pipe radius, L_2 = length of plate

for $P_0 = 14.0$ $t_m = 0.57$ $(P = 900 \text{ psi})$
 12.5 $t_m = 0.59$

$$r = (P_0 - t_m) / 2 = 6.710$$

$$= 6.030$$

Node	L_1	β_1	r	L_2	β_2	$\beta_1 \beta_2$	
100	1 1/4	0.19	6.71	2 1/2	0.59	0.111	NG
135	1 1/4	0.19	6.71	2 1/2	0.59	0.111	NG
170	2	0.30	6.71	1 1/2	0.30	0.089	NG
	1 1/4	0.19	6.71	2 1/2	0.59	0.111	NG
260	1 1/4	0.19	6.71	2 1/2	0.59	0.111	NG
380	1 1/4	0.19	6.71	2 1/2	0.59	0.111	NG
485	1	0.17	6.03	2 1/2	0.59	0.173	NG
590	1	0.17	6.03	2 1/2	0.59	0.173	NG
855	1	0.17	6.03	2 1/2	0.59	0.173	NG

100: try $L_2 = 0.5r = 3.35$, but $\beta_1 \beta_2 = 0.095 > 0.075$
 so $L_2 = 0.075r / \beta_1 = 2.65$

135: same as 100

170: use $L_2 = 0.075r / \beta_1 = 1.68$ for 2" L_2
 same as 100 for 2" L_2

260: same as 100

380: " " "

485: try $L_2 = 0.5r = 3.35$, $\beta_1 \beta_2 = 0.075$ ok

855: use 485 results

490: use $L_2 = 0.5r = 3.35$, $\beta_1 \beta_2 = 0.085$ NG.
 try $L_2 = 0.075r / \beta_1 = (0.075)(6.03) / 0.17 = 2.65$

L465

$$1.4 \quad \sqrt{r \pm} = \sqrt{(6.71)(0.57)} = 1.97'' \rightarrow 14.5/1.97 = 22 \text{ evenly spaced allowed: OK}$$

$$1.5 \quad D_o/\pm = 14/0.57 = 25 < 100$$

$$2.0 \quad r = 6.71''/6.03''$$

$$\pm = 0.57''/0.69''$$

$$D_o = 14''/12.75''$$

$$T = r/\pm = 11.569/9.779$$

$$b_1, b_2 \text{ from p1}$$

$$L_1, L_2 \text{ from p1}$$

$$K_1 = 3.6 \text{ for filled web}$$

$$M_L = M_N = M_T = Q_1 = W = 0$$

$$Q_2 \neq 0, \text{ load}$$

3.0

$$S_{m1} = Q_2 / 2L_2L_6$$

$$L_6 = \min(L_1, \pm) = \min(L_1, 0.57) \text{ from } 14'', \text{ or } \min(L_1, 0.69) \text{ from } 12''$$

Node	Q_2	L_2	L_6	S_{m1}
100	12.1	2.65	0.57	7.94 ksi
175	29.7	2.65	0.57	9.74
170	17.7	1.68	0.57	7.03
	17.7	2.65	0.57	4.46
260	31.4	2.65	0.57	10.21
380	47.4	2.65	0.57	15.12
485	41.0	2.66	0.69	11.17
480	14.5	2.66	0.69	3.95
855	9.5	2.66	0.69	2.06

$$S_{m1} = Q_2 / 2L_2L_6 = S_{m2}$$

$$S_{m1} \neq S_{m2}$$

7.0 - for eqn 7 & special eqns only.

$$(NC-9) \quad \underbrace{\frac{B_1 P_{avg} P_o}{2t_n} + B_2 \frac{M_A + M_B}{L}}_{\sigma_p} + s_{ml} \leq \min(2.5 s_y, 2 s_y)$$

σ_p is the standard piping output stress. for straight pipe, the 0.75 factor of the equation which will be used here, gives identical results

for T_p 716 @ 120°F,

$$s_m = 18,500 \text{ psi}$$

$$s_y = 27,536 \text{ psi}$$

$$\min(2.5 s_y, 2 s_y) = 564 \text{ (i.e. } 3 s_y)$$

Node	σ_p	s_{m1}	$\sigma_p + s_{mL}$	$< 56.4 \text{ ksi?}$	$s_{mL} \text{ 418 ksi?}$
100	8.8	3.44	12.7	OK-0.22	OK 0.21
135	9.5	4.74	23.2	OK 0.41	OK 0.50
170	9.1	7.03	16.1	OK-0.29	OK 0.77
170	9.1	4.76	13.6	OK-0.24	OK 0.24
260	17.5	10.21	29.3	OK 0.52	OK 0.21
380	12.6	15.42	29.6	OK 0.52	OK 0.82
475	12.8	16.17	30.0	OK-0.53	OK 0.59
490	14.3	3.95	18.3	OK 0.32	OK 0.21
555	13.5	2.06	15.6	OK 0.28	OK 0.11

$$s_{mL}^{**} \leq 2.5 s_y \Rightarrow \text{since } s_{m1} = s_{mL} = s_{mL} < 2.0 s_y, \text{ satisfied by the above}$$

$$Q_2^{**} / 2L_2 \leq s_y :$$

since $Q_2^{**} = Q_2$, this is equivalent to

$$s_{mL} < s_y$$

3.2 - ~~with~~ require for fillet welds.

$$(7) \quad \frac{2Q_2^{**}}{A_w} \leq 2s_y \leq 56.4 \text{ ksi}$$

$Q_2^{**} = Q_2$, A_w = fillet weld throat area

$$A_w = t (n_1 L_1 + n_2 L_2)$$

t = throat size

n_1 = # L_1 sides welded

n_2 = # L_2 " "

$$(8) \quad \{4 [Q_2^{**}/A_w]^2\}^{1/2} \leq s_y$$

since $Q_2^{**} = Q_2$, this reduces to

$$2Q_2/A_w \leq s_y$$

if this is satisfied, (7) is automatically satisfied

node	Q_2	n_1	L_1	n_2	L_2	A_w	$2Q_2/A_w$	$< s_y = 18.8?$
100	12.1	1	6.25	2	2.65	6.55	3.7	OK - 0.20
135	28.7	1	1.25	2	2.45	6.55	8.76	OK - 0.47
171	13.7	1	3.0	2	1.68	5.36	5.11	OK - 0.27
	17.7	1	1.25	2	2.65	6.55	4.18	OK 0.22
260	31.4	1	1.25	2	2.65	6.55	9.59	OK 0.51
380	47.4	1	1.25	2	2.65	6.55	7.24	OK 0.39
485	41.0	1	1.20	2	3.35	7.70	10.65	OK 0.57
490	14.5	1	1.0	2	3.06	7.91	5.90	OK 0.31
885	9.5	1	1.0	2	3.35	7.70	2.47	OK 0.13

INEL

4/12/86

SONBSS Confirm Anal

MGR

Spring hanger table

Support No.	Node/dp	Type [Grinnell]	Loads		Stress Ratio
			Hot	Max	
SI-05-0720-H014	25/35	Fig 8-258, #16, 'B'	6500	9750	0.67
SI-05-6004-H010	85/65	" #16, 'B'	4491	5200	0.86
SI-06-0319-H004	475/390	Fig 268, #17, 'B'	10100	13000	0.78
SI-06-0319-H005	675/375	" #10, 'F'	1458	1690	0.86
SI-05-0720-H017	855/310	Fig 82, #17, 'F'	4500	5200	0.87

Stress Ratio = $\frac{\text{hot load}}{\text{max load}}$

Max Loads taken From ITT Grinnell PH 71 catalog

ST-51 Inway Street Analysis

12/12/95

NOV / DP	Support No.	Length	Load	Part No. / size	Allowable		Comment
					length	load (normal)	
135/15	SI-05-6004-11014	3' 8 3/4"	12527	2200-30	78"	30000	OK-0.42
180/115	SI-02-6004-11002	4' 10 3/4"	10556	C-211/421	120"	15400	(1) OK-0.69
145/120	SI-02-6004-11006	5' 7"	9216	211/42	120"	11630	OK - 0.79
200/125	SI-02-6004-11001	4' 10 3/4"	11840	C-211/421	120"	15400	(1) OK-0.77
260/157	SI-02-6005-11006	4' 1"	14751	2200-30	78"	30000	OK - 0.49
235/175	SI-02-6005-11005	4' 5 1/2"	5500	211/42	120"	11630	OK - 0.47
305/175	SI-02-6005-11002	2' 4 1/2"	9204	C-211/42	120"	11630	OK - 0.80
330/205	SI-02-6005-11008	3' 9"	13307	2200-30	78"	30000	OK - 0.44

7/12/26

SWAY STRUT NOTES

- 2) The given allowable for these struts is for a length of 10', where buckling would clearly govern. Because of this, a buckling calculation will be used to establish capacity. This is attached.

mm 2/12/55

SWAY TRUT ANALYSIS

Struts @ nodes 180 & 200 are C-211, size 2 Channel sway struts 4'10" long loaded to 10556 and 16, respectively. the capacity of these supports will be assessed in terms of the buckling load, and shear of the bolts.

Buckling - based on a pinned-pinned, 2" x 160 pipe, 4'10" long.

$$2" \text{ x } 160 \text{ pipe: } A = 2.190 \text{ in}^2 \\ I = 1.162 \text{ in}^4$$

$$r = \sqrt{I/A} = \sqrt{\frac{1.162}{2.190}} = 0.728 \text{ in}$$

$$K = 1.0 \text{ (pinned-pinned, } P = 124, A = 50, 330)$$

$$\frac{KL}{r} = \frac{58}{0.728} = 79.625 < 200$$

Assume $F_y = 36 \text{ ksi}$

$$C_c = \sqrt{\frac{2\pi^2(29000)}{(36/0.728)^2}} \quad 1.5-1, A = 50, 330$$

$$C_c = 126.1$$

$$KL/r < C_c:$$

$$F_a = \frac{1 - \frac{79.625^2}{2(126.1)^2}}{5} \cdot 36 \\ \frac{5}{8} = \frac{3(126.1)^2}{5(126.1)^2} - \frac{(79.625)^2}{8(126.1)^2}$$

$$F_a = 15.797$$

strut @ node 180, blank, @ 200, OK

note: use 1.77 seismic factor, $F_a = 1.77(15.797) = 27.87$

$F_a < F_y$ @ node 80

Model/Sp	Length	Load (lb)	Support No.	Manuf.	Model No.	Allow. Length	Load Rating	Comments
22/30	15'	1193	SI-05-0100-H013	Pacific Scientific	PSA-3	4.5'	11,700	OK
100/75	4.5'	5018	SI-05-6004-H012	Pacific Scientific	PSA-10	4.6'	72,100	OK
170/100	3.8'	12673	SI-02-6004-H005	Pac. Sci.	PSA-10	4.6'	22,100	OK-059
410/196	7.6'	13605	SI-02-6005-H001	Pac. Sci.	PSA-15	10.0'	72,450	OK
310/110	8.4'	16713	SI-06-6005-H001	Pac. Sci.	PSA-15	10.0'	72,450	OK
315/225	3.0'	110189	SI-06-1005-H005	Pac. Sci.	PSA-10	4.6'	22,100	N6-1.82
380/270	5.2'	18787	SI-06-6005-H001	Pac. Sci.	PSA-10	4.6'	17,114 (1)	N6-1.11
410/537	3.2'	18970	SI-06-0119-H007	Pac. Sci.	PSA-10	10.0'	72,450	OK
470/538	6.5'	11053	SI-06-0119-1/006	Pac. Sci.	PSA-15	10.0'	72,450	OK
535/570	3.8'	10596	SI-06-0123-H006	Pac. Sci.	PSA-15	10.0'	72,450	OK
625/705	4.0'	17552	SI-04-0119-H010	Pac. Sci.	PSA-15	10.0'	72,450	OK
846/107	6.5'	6162	SI-05-6004-H009	Pac. Sci.	PSA-10	4.6'	15,459 (2)	0.11-OK
835/512	5.5'	7251	SI-05-0120-H016	Pac. Sci.	PSA-15	10.0'	72,450	OK
861/	5.6'	6206	SI-05-012-H011	Pac. Sci.	PSA-15	10.0'	72,450	OK
870/120	7.2'	11553	SI-05-0110-H009	Pac. Sci.	PSA-15	10.0'	72,450	OK

II-51 Snubber Table Notes

- 2) The buckling load, away from simple compression failure (short columns), is controlled by the following relationships:

$$P_1 = K_1 / L_1^2$$

$$P_2 = K_2 / L_2^2$$

for the same snubber $K_1 = K_2$:

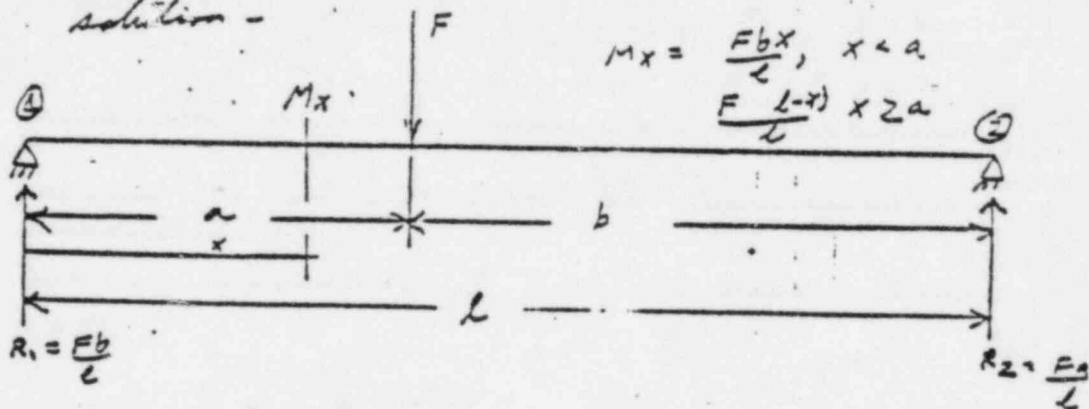
$$P_1 / P_2 = (L_2 / L_1)^2$$

$$P_1 = (22100) (4.6 / 5.2)^2 =$$

- 2) Same as above:

$$P_1 = (22100) (4.6 / 5.5)^2 = 15457$$

SS solution -



given: L, a, F, I, S

wanted: $R_1, R_2, \bar{F}_b(x)$

Register	Contents
0	L
1	a
2	F
3	
4	S
5	b
6	x

PRP *SS*

01 *LBL *SS*

02 XEQ *IMP*

03 RCL 02

04 RCL 05

05 *

06 RCL 09

07 /

08 *R1= *

09 APCL X

10 PROMPT

11 RCL 02

12 RCL 01

13 *

14 RCL 09

15 /

16 *R2= *

17 APCL X

18 PROMPT

19 RCL 02

20 RCL 05

21 *

22 RCL 01

23 *

24 RCL 09

25 /

26 RCL 04

27 /

28 *R3= *

29 APCL X

30 PROMPT

31 RCL 01

01 *LBL 01

02 * L=?*

03 PROMPT

04 STO 06

05 RCL 01

06 XNY?

07 GTO 02

08 RCL 09

09 RCL 06

10 -

11 RCL 02

12 *

13 RCL 01

14 *

15 RCL 09

16 /

17 GTO 03

18 *LBL 02

19 RCL 02

20 RCL 05

21 *

22 RCL 06

23 *

24 RCL 09

25 /

26 *LBL 03

27 RCL 04

28 *

29 *R3= *

30 APCL X

31 PROMPT

32 STO 01

33 END

PRP *IMP*

01 *LBL *IMP*

02 * L=?*

03 PROMPT

04 STO 09

05 * A=?*

06 PROMPT

07 STO 01

08 * F=?*

09 PROMPT

10 STO 02

11 * S=?*

12 PROMPT

13 STO 04

14 RCL 09

15 RCL 01

16 -

17 STO 05

18 END

10000

4

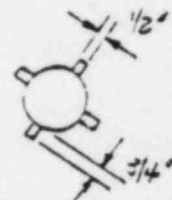
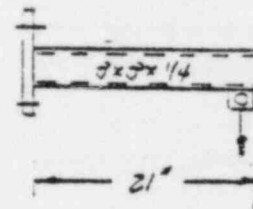
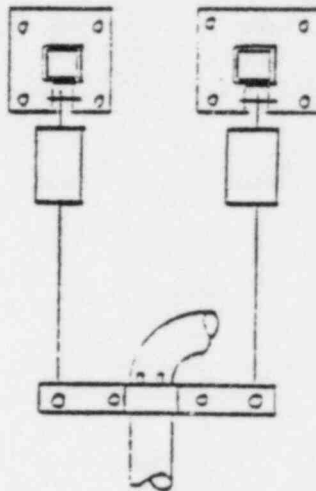
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D-66

JZ-01-6007-H00A

LOADS:

$$|W+P_d| = 1050 \text{ lb.}$$



CONSIDER LUGS: $1/2" \times 3/4" \times 1"$ Long

$$\text{Stress: } \sigma_v = \frac{1050}{4(1/2)(1)} = 525 \text{ PSI} < 14,400 \text{ PSI}$$

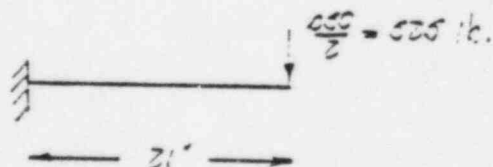
CONSIDER SPRING HANGER & CLEVISES:

Assume Grinnell parts -

<u>Part</u>	<u>Allowable Load (lb.)</u>
290	1810
140	1810
66	1810
(2) 8-268, TO. C	1064
Size 7	

Actual loads are less than allowable.

CONSIDER 3x3x1/4 TUBING: (use longest beam)



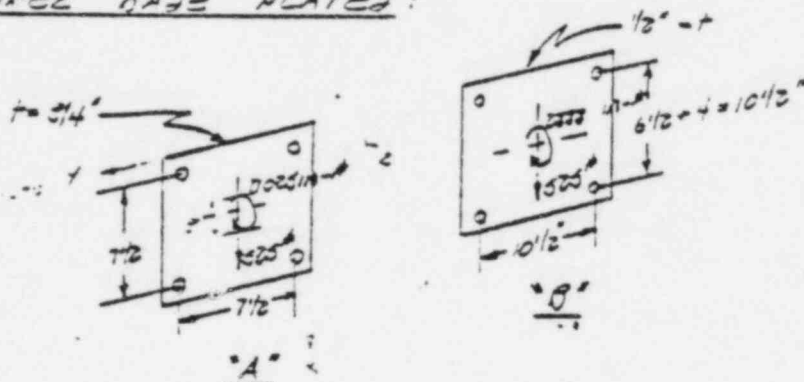
$$A = 2.52 \text{ in}^2$$

$$S = 2.10 \text{ in}^3$$

shear: $\tau_v = \frac{525}{2.59} = 203 \text{ psi} < 14,400 \text{ psi}$

bending: $\sigma_b = \frac{525(21)}{2.10} = 5250 \text{ psi} < 23,760 \text{ psi}$

CONSIDER BASE PLATES:



Use "PLATE" program —

bending: $\sigma_{b_{max}} = 12,882 \text{ psi}$ For plate "A" $< 23,760 \text{ psi}$

$\sigma_{b_{max}} = 8,697 \text{ psi}$ For plate "B" $< 23,760 \text{ psi}$

tension: $\tau_t = \frac{525}{[9 - 2(3/4)](3/4)} = 93 \text{ psi} < 21,600 \text{ psi}$

shear: $\tau_v = \frac{525}{4(1 1/2)(1/2)} = 175 \text{ psi} < 14,400 \text{ psi}$

CONSIDER BOLTS:

For plate "A" →

tension: $P = \frac{11025}{(7.5)(2)} = 735 \text{ lb/bolt}$

shear: $P = \frac{525}{4} = 131 \text{ lb/bolt}$

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For plate "B" →

Tension: $P = \frac{2222}{(10/2)(2)} = 111 \text{ lb/bolt}$

shear: $P = \frac{525}{4} = 131 \text{ lb/bolt}$

These loads appear reasonable for the respective bolt sizes, however, allowables are unknown.

CONSIDER WELDS:

Lugs to pipe → 7/16" fillet, both sides

shear: $\sigma_v = \frac{525}{.707(7/16)(8)(1)} = 495 \text{ psi}$

Clevis to tubing → 1/4" fillet, both sides

Tension: $\sigma_t = \frac{525}{.707(1/4)(2)(11/4)} = 1188 \text{ psi}$

Tubing to plate → 1/4" fillet, all around

shear: $\sigma_v = \frac{525}{.707(1/4)(6)} = 495 \text{ psi}$

Bending: $\sigma_b = \frac{525(21)}{.707(1/4)(6)(3)} = 2,465 \text{ psi}$

CONSIDER RIGID CLAMP:

For a Grinnell H740 clamp, allowable load
= 3450 lbs > 1050 lbs.

∴ This support is acceptable.

PLATE "A"

03	2.25	2.25	
04	3.75	3.75	
05	2.25	5.25	
06	3.75	3.75	
07	7.5	7.5	
08	7.5	7.5	
09	3675	-3675	
10	6.52E-10	6.52E-10	
14	1.12E+06	1.12E+06	
11	Wxx	-0.0007	.00003
12	Wyy	-0.0007	.00019
13	Wxy	-4.72E-15	3.52E-15
14	Mxx	1046.55	-97.86
15	Myy	984.07	-223.60
16	Mxy	-3.70E-09	2.76E-09
			<u>TOTAL</u>
18	Tx	11,163.22	-1043.79
19	Ty	10,496.72	-2335.04
20	Txy	-3.95E-08	2.95E-08
			12,207
			12,832
			6.90E-08

PLATE "B"

03	3.75	3.75
04	5.25	5.25
05	3.75	6.75
06	5.25	5.25
07	10.5	10.5
08	10.5	10.5
09	963	-963
10	1.12E-09	1.12E-09
14	3.32E+05	3.32E+05

w_{xx}	- .0007	.00007
w_{yy}	- .0006	.0003
w_{xy}	-1.14E-14	-2.64E-14

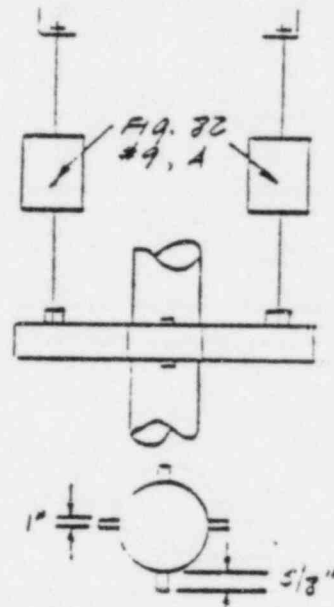
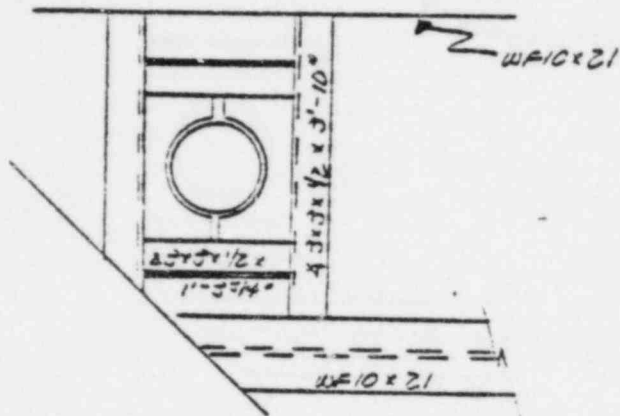
m_{xx}	284.50	-46.87
m_{yy}	272.44	-89.95
m_{xy}	-2.65E-09	-6.14E-09

TOTAL

σ_x	6,827.90	-1,124.83	7,952.73
σ_y	6,558.67	-2,158.73	8,697.40
σ_{xy}	-6.3E-08	-1.47E-07	2.10E-07

SI-01-6007-H000

Loads: $|W + P_d + T| = 2040 \text{ lbs.}$



CONSIDER LUGS: $5/8" \times 1" \times 1"$ long

Shear: $\sigma_v = \frac{2040}{4(1)(1)} = 510 \text{ psi} < 14,400 \text{ psi}$

CONSIDER CLAMP:

For a Grinnell H540 clamp with $5/8"$ bolts,
allowable load = 3450 lbs $>$ 2040 lbs.

CONSIDER SPRING HANGERS & CLEVISES:

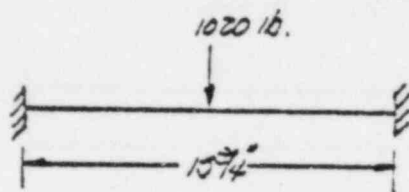
Assume Grinnell parts —

Part	Allowable Load (lbs)
290	3710
140	3710
(2) 82, Type A #9	2040/2042

Actual loads $\leq$ allowables

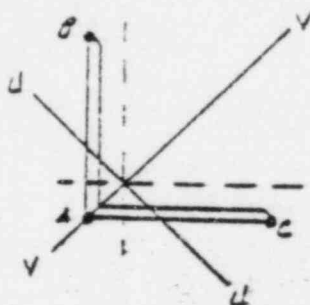
CONSIDER $2 \times 3 \times \frac{1}{2} \times 1' - \frac{3}{4}"$:

Consider as a fixed-fixed beam w/ loads applied at center —



Shear: $\sigma_v = \frac{1020/2}{2.75} = 185 \text{ psi} < 14,400 \text{ psi}$

Bending: $a = 2.75 \text{ in}^2$ $I_x = I_y = 2.22 \text{ in}^4$
 $x_o = y_o = 0.952 \text{ in}$ $r_u = 0.584 \text{ in}$



$$I_U = 2.75 (.584)^2 = 0.938 \text{ in}^4$$

$$I_V = 2(2.22) - .938 = 3.502 \text{ in}^4$$

$$v_A = -1.318$$

$$v_B = 1.478$$

$$v_C = 1.478$$

$$u_A = 0.$$

$$u_B = -2.796$$

$$u_C = 2.796$$

$$m_u = \frac{-707(1020)(15\frac{3}{4})}{8} = -1420 \text{ in-lb}$$

$$m_v = \frac{707(1020)(15\frac{3}{4})}{8} = 1420 \text{ in-lb.}$$

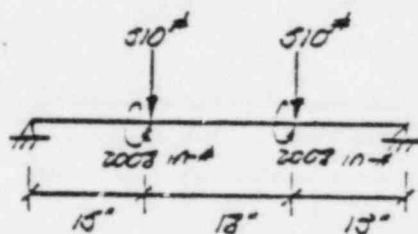
$$\sigma_A = \frac{-1420(-1.318)}{.938} = 1995 \text{ psi} < 21,600 \text{ psi}$$

$$\sigma_B = \frac{-1420(1.478)}{.938} + \frac{1420(-2.796)}{3.502} = 3371 \text{ psi} < 21,600 \text{ psi}$$

$$\sigma_C = \frac{-1420(1.478)}{.938} + \frac{1420(2.796)}{3.502} = -1104 \text{ psi} < 21,600 \text{ psi}$$

CONSIDER $2 \times 3 \times 3 \times 1/2 \times 3'-10"$:

consider as a fixed-fixed beam with loads applied as shown —



shear:
$$\sigma_v = \frac{\frac{2(510)(31)^2[(3)(15) + 31]}{46^3}}{2.75} = 223 \text{ psi} < 14,400 \text{ psi}$$

bending:
$$m_{max} = \frac{2(510)(15)(31)^2}{46^3} = 6949 \text{ in-lb}$$

$$m_u = -.707(6949) = -4913 \text{ in-lb}$$

$$m_v = .707(6949) = 4913 \text{ in-lb}$$

$$\sigma_A = \frac{-4913(-1.318)}{.938} = +6903 \text{ psi} < 21,600 \text{ psi}$$

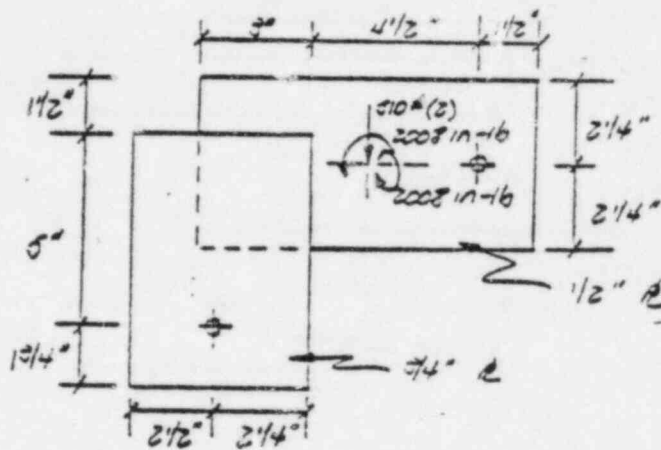
$$\sigma_B = \frac{-4913(1.473)}{.938} + \frac{4913(-2.795)}{3.502} = 11,664 \text{ psi} < 21,600 \text{ psi}$$

$$\sigma_C = \frac{-4913(1.473)}{.938} + \frac{4913(2.795)}{3.502} = -3819 \text{ psi} < 21,600 \text{ psi}$$

torsion:
$$J = \frac{1}{3} \left[3(1/2)^3 + (3-1/2)(1/2)^3 \right] = 0.229 \text{ in}^4$$

$$\tau = \frac{T}{J} = \frac{2(2008)(1/2)}{.229} = 8769 \text{ psi} < 14,400 \text{ psi}$$

CONSIDER PIPE PLATE:



$$419 \text{ lb.} = \frac{2(310)(2)}{2.75} = \frac{2008}{2.75}$$

$$324 \text{ lb.} = \frac{2008}{2.75} + \frac{2(310)(2)}{2.75} = 324 \text{ lb.}$$

For 1/2" R -

shear: $\sigma_v = \frac{324}{(4 1/2 - 3/4)(1/2)} = 445 \text{ psi} < 14,400 \text{ psi}$

Torsion: $T = \frac{2008(1/2)(2)}{4 1/2 (1/2)^2} = 5355 \text{ psi} < 14,400 \text{ psi}$

For 3/4" R -

shear: $\sigma_v = \frac{419}{2(1 3/4)(3/4)} = 160 \text{ psi} < 14,400 \text{ psi}$

Tension: $\sigma_t = \frac{419}{(4 3/4 - 1)(3/4)} = 150 \text{ psi} < 21,600 \text{ psi}$

Bending: $\sigma_b = \frac{2008(3/4)(2)}{4 3/4 (3/4)^2 / 12} = 4509 \text{ psi} < 27,000 \text{ psi}$

CONSIDER BOLTS:

For 3/4" Fig. 5-34 Philips -

$$\text{shear} = 834 \text{ lb} < 16,516/5 = 3303 \text{ lb. allowable}$$

$$\text{tension} = \frac{2003}{4.25} = 472 \text{ lb.} < 17,675/5 = 3535 \text{ lb. allow}$$

For 1" Hitch bolt -

$$\text{shear} = 419 \text{ lb.} < 26,879/5 = 5376 \text{ lb. allowable}$$

$$\text{tension} = 472 \text{ lb.} < 14,000/5 = 2800 \text{ lb. allow.}$$

CONSIDER WELDS:

$\Delta 3 \times 3 \times 1/2$ to $\Delta 3 \times 3 \times 1/2$ - 3/16" Fillet

$$\text{shear: } \sigma_v = \frac{1020/2}{.707 (3/16)(12)} = 321 \text{ psi}$$

$$\text{bending: } \sigma_b = \frac{1020 (15 \cdot 3/4) / 2}{.707 (3/16) \left[\frac{4(3)(15) + (3)^2}{6} \right]} = 2020 \text{ psi}$$

$$\text{Total: } 2341 \text{ psi}$$

$\Delta 3 \times 3 \times 1/2$ to anchor - 3/16" Fillet

$$\text{shear: } \sigma_v = \frac{\frac{2(510)(31)^2}{4 \cdot 23} [3(15) - 31]}{.707 (3/16)(12)} = 786 \text{ psi}$$

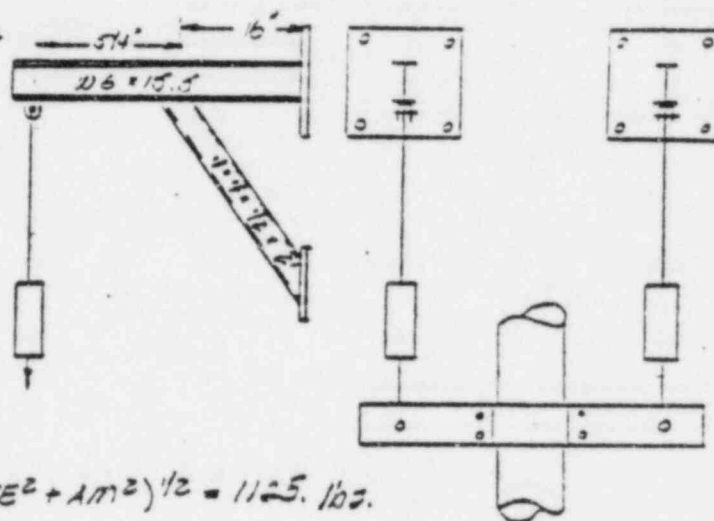
$$\text{bending: } \sigma_b = \frac{6949}{.707 (3/16) \left[\frac{4(3)(31) + (3)^2}{6} \right]} = 6939 \text{ psi}$$

$$\text{Torsion: } T = \frac{2(2003)}{.707 (3/16) \left[\frac{(3+31)^2 - 3(3)^2/2}{12(3+31)} \right]} = 2693 \text{ psi}$$

$$\text{Total: } 10,068 \text{ psi}$$

$\therefore$ This support appears acceptable.

SI-01-6007-HCOC



LOADS:

$$|W + P_d| + (I E^2 + A M^2)^{1/2} = 1125 \text{ lbs.}$$

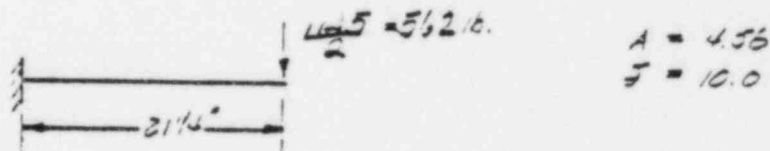
CONSIDER SLURGED AND CLEVERED:

FOR PJA-3, allowable load = 6000 lbs. LEVEL A = 0
 = 10,780 lbs. LEVEL 3 = 0

∴ Imposed load is below allowable

CONSIDER HORIZONTAL WIDE FLANGE:

conservatively consider WF as a cantilever beam
 w/out bracing -



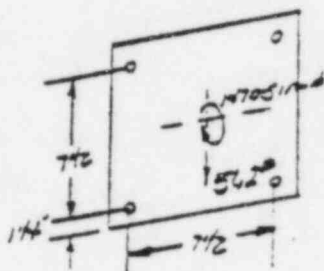
shear: $\sigma_v = \frac{562}{4.56} = 123 \text{ psi} < 14,400 \text{ psi}$

bending: $\sigma_b = \frac{562 (21' 1/4)}{12} = 1194 \text{ psi} < 27,750 \text{ psi}$

CONSIDER LUGS: $3/4" \times 1" \times 2"$ LONG

$$\text{shear: } \sigma_v = \frac{1125}{4(1)(2)} = 141 \text{ psi} < 14,400 \text{ psi}$$

CONSIDER BASE PLATES: $3/4" \times 10" \times 10"$



Bending: Conservatively consider plate as a simply supported beam with moment applied at center -

$$\sigma_b = \frac{11943 (3/4 / 2)}{7/2 (3/4)^3 / 12} = 16986 \text{ psi} < 23,760 \text{ psi}$$

$$\text{shear: } \sigma_v = \frac{562}{4(1/4)(3/4)} = 150 \text{ psi} < 14,400 \text{ psi}$$

$$\text{TENSION: } \sigma_t = \frac{562}{[10 - 2(3/4)](3/4)} = 88 \text{ psi} < 21,600 \text{ psi}$$

CONSIDER BOLTS:

$$\text{TENSION: } R_t = \frac{11943}{7/2 (3)} = 794 \text{ lb / bolt}$$

$$\text{shear: } R_s = \frac{562}{4} = 140 \text{ lb / bolt}$$

Although the type of anchor bolt is uncertain, these loads appear reasonable for $3/4"$ bolts.

CONSIDER WELDS:

CLAVIS to WF $\rightarrow$ $\frac{1}{4}$ " Fillet, all around

$$\text{Tension: } \sigma_t = \frac{562}{.707(\frac{1}{4})(2.25)(4)} = 353 \text{ psi}$$

WF to plate $\rightarrow$ $\frac{1}{4}$ " Fillet, all around

$$\text{Shear: } \tau_v = \frac{562}{2(5.45)(.707)(\frac{1}{4})} = 291 \text{ psi}$$

$$\text{Bending: } \sigma_b = \frac{11943}{.707(\frac{1}{4})(6)(5.995)} = 1879 \text{ psi}$$

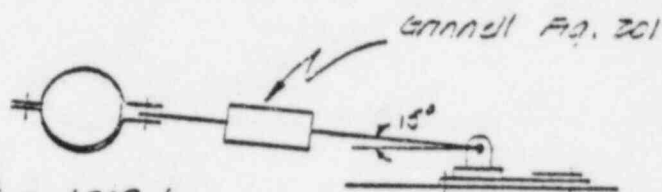
CONSIDER RISSER CLAMP:

Assume 2 6" Grinnell MS 40 riser clamp:

$$\text{Allowable load} = 3450 \text{ lbs.} > 1125 \text{ lbs.}$$

$\therefore$ This support is acceptable.

J2-01-5007-HOOD



LOADS: $|W + P_d| + (33E^2 + 4M^2)^{1/2} = 1008 \text{ lbs.}$

CONSIDER SHURTER, CLAMP, CLEVIS:

For Grinnell Fig. 201 assembly, the allowable load = 10,350 lbs. > 1008 lbs.

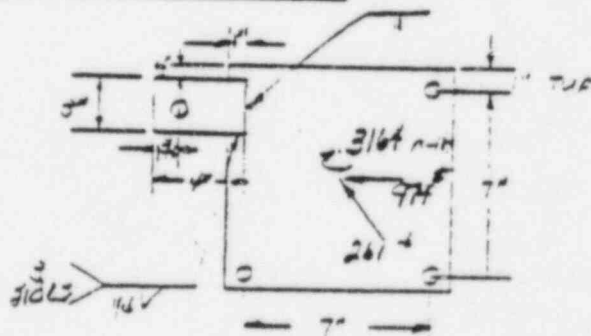
CONSIDER WELDS:

Clevis plate to base plate — $\frac{3}{16}$ " fillet all around

Shear: $\sigma_v = \frac{1008 \text{ lbs.}}{.707(\frac{3}{16})(10)} = 441 \text{ psi}$

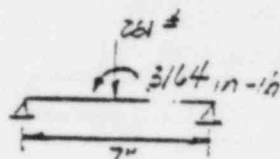
Tension: $\sigma_t = \left[\frac{(1008 \text{ lbs.})(\frac{3}{4})}{5} + 1010 \sin 15^\circ \right] \frac{1}{.707(\frac{3}{16})(10)}$
 $= 405 \text{ psi}$

CONSIDER CASE PLATE + BOLTS:



1. First, consider main plate as is. Here we have a rigid connector in upper left corner.

Bending:

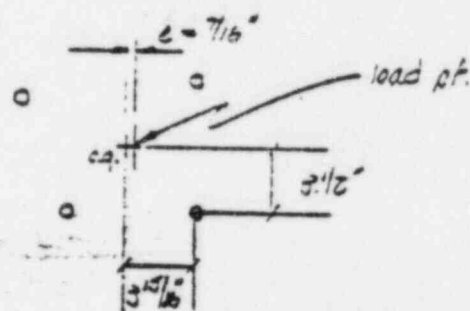


$$\sigma_b = \left[\frac{3164 + \frac{251(7)}{4}}{7 \left(\frac{1/2 \cdot 7^3}{12} \right)} \right] = 12414 \text{ psi} < 23,760 \text{ psi}$$

Tension: $\sigma_t = \frac{974}{[7 - 2(7/4)] \cdot 1/2} = 260 \text{ psi} < 21,600 \text{ psi}$

Shear: $\sigma_v = \frac{974}{4(1)(1/2)} = 487 \text{ psi} < 14,400 \text{ psi}$

2. Second, consider bolt loads taking into account eccentric bolt pattern:



Since c.g. of bolt pattern is so close to point of loading, ignore eccentricity and consider safety factor in doing so.

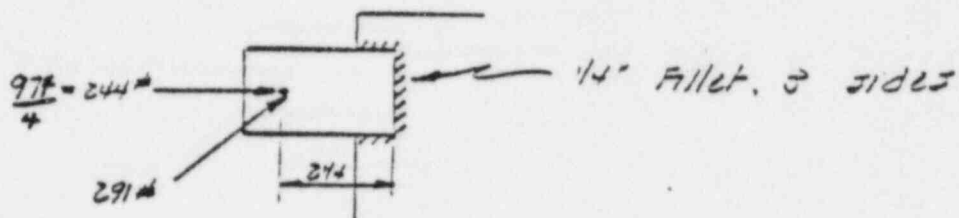
Shear: $\sigma_v = \frac{974}{4} = 244 \text{ lb/bolt}$

Tension: $\sigma_t = \frac{251}{4} + \frac{3164}{2(7)} = 291 \text{ lb/bolt}$

For 3/4" High Strength Bolts the ultimate shear load is 13,257 lb, and the ultimate tensile load is 2155 lb.

Although the allowable loads for the 3/4" bolts are unknown, the imposed loads are so small that they are deemed acceptable.

3. Consider joint of extension plate to main plate:



$$\sigma = \left[\frac{244 + 291 + \frac{291(244)}{1^2[(2)(3)+1]/5(1+3)}}{5} \right] \frac{1}{.707(1/4)} = 6956 \text{ psi}$$

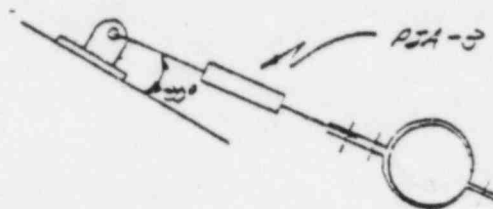
$\therefore$ This support is acceptable.

NATIONAL ASSOCIATION OF BROADCASTERS
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 www.nab.org

LC405:

$$|\omega + P_d + T| + (J^2 E^2 + 1m^2)^{1/2}$$

$$= 862 \text{ Ib}.$$

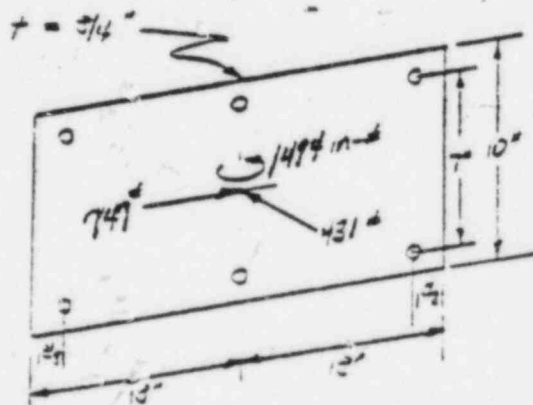


CONSIDER INJURED, CLAMP, CLEVIS:

Assume all Pacific Scientific PSA-3 parts:

Allowable load = 6000 lbs Level A = 0
10,580 lbs Level C = 0

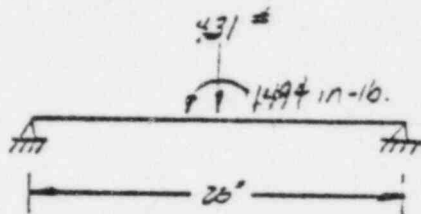
CONSIDER CASE PLATE:



Tension: $\sigma_f = \frac{747}{[10 - 2(7/16)](3/4)} = 119 \text{ psi} < 21,600 \text{ psi}$

Test: $T_1 = \frac{747}{4(178)(7/4)} = 181 \text{ psi} < 14,400 \text{ psi}$

Findings: Conservatively, more inside bolts and
wider plate as a simply supported
beam with moment and pt. load
applied at center -



$$\sigma_b = \frac{mc}{I} = \left[\frac{431(26)}{4} + 1494 \right] \frac{(3/4/2)}{7(3/4)^3/12} = 6546 \text{ psi} < 23,750 \text{ psi}$$

CONSIDER BOLTS:

shear: $\frac{747}{6} = 125 \text{ lb/bolt}$

TENSION: consider corner bolts only -

$$\frac{1494}{2(26)} - \frac{431}{4} = 136 \text{ lb/bolt}$$

$\therefore$ Although bolt type is unknown, these loads appear acceptable for 3/4" concrete anchors.

CONSIDER WELDS:

clevis plate to base plate - 1/4" fillet all around

shear: $\sigma_v = \frac{747}{.707(1/4)(10)} = 423 \text{ psi}$

TENSION: $\sigma_t = \frac{431 + 1494/5}{.707(1/4)(10)} = 413 \text{ psi}$

$\therefore$ This support is acceptable.

SI-01-6007-H20F

114" x 4" x 6" 1002

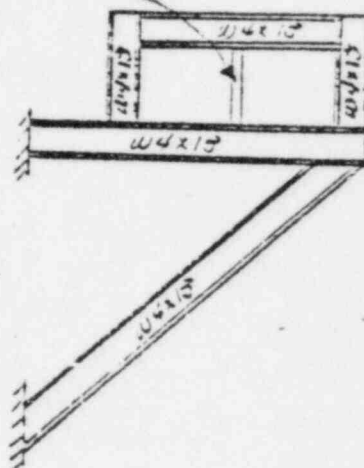
LOADS:

Y-dir. $|W + P_d + T| = 1036 \text{ lbs}$

$$|W + P_d + T| + (JSE^2 + Am^2)^{1/2} = 1645 \text{ lbs.}$$

INC. $|W + P_d + T| = 103 \text{ lbs.}$

$$|W + P_d + T| + (JSE^2 + Am^2)^{1/2} = 442 \text{ lbs.}$$



Utilize computer program "FRAME" to find reactions in structural members: Analyze for worst load combinations

CONSIDER W4x13 MEMBERS:

$$A = 3.83 \text{ in}^2, J = 5.46 \text{ in}^4$$

TENSION: $\sigma_t = \frac{1825}{3.83} = 477 \text{ psi} < 21,600 \text{ psi}$

COMPRESSION: $C_c = 126$

$$\frac{KL}{r} = \frac{2(42)}{1.0} = 84 < 126$$

$$F_a = \left[\frac{1 - \frac{(84)^2}{2(126)^2}}{\frac{5}{3} + \frac{2(84)}{3(126)} - \frac{(84)^2}{3(126)^2}} \right] 21,000 = 14,897 \text{ psi}$$

LEVEL D = 23010 psi

$$\sigma_a = \frac{1824}{3.83} = 479 \text{ psi} < 14,897 \text{ psi}$$

shear: $\tau_v = \frac{1162}{3.83} = 303 \text{ psi} < 14,400 \text{ psi}$
 $\rho = 85\%$

Bending: $\sigma_b = \frac{3964}{5.46} = 726 \text{ psi} < 23,760 \text{ psi}$

CONSIDER 1" x 4" x 6 A:

Tension: $\sigma_t = \frac{576}{5.0} = 117 \text{ psi} < 21,600 \text{ psi}$

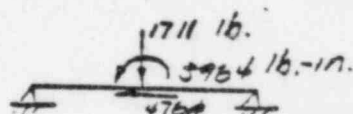
Shear: $\tau_v = \frac{266}{5} = 53 \text{ psi} < 14,400 \text{ psi}$

Bending: $\sigma_b = \frac{789 (1/4 / 2)}{4(1/4)^3 / 12} = 757 \text{ psi} < 27,000 \text{ psi}$

CONSIDER CASE PLATES:

Analyze plates as simply supported beam with loads applied at center:

Top plate:

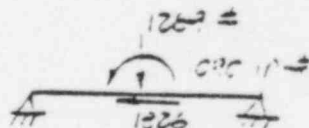


Bending: $\left[3964 + \frac{1711(20)}{4} \right] \frac{(1/2)}{17(1)^3 / 12} = 4413 \text{ psi} < 27,000 \text{ psi}$

Tension: $\sigma_t = \frac{476}{[21 - 2(1^3/8)](1)} = 26 \text{ psi} < 21,600 \text{ psi}$

Shear: $\tau_v = \frac{476}{4(2)(1)} = 60 \text{ psi} < 14,400 \text{ psi}$

Bottom plate:



D-86

$$\text{Bending: } \left[1090 + \frac{1269(13)}{4} \right] \frac{(1/2/2)}{13(1/2)^2/12} = 9626 \text{ psi} < 27000 \text{ psi}$$

$$\text{Tension: } \sigma_t = \frac{1326}{[16 - 2(13/16)]^{1/2}} = 184 \text{ psi} < 21,600 \text{ psi}$$

$$\text{Shear: } \sigma_v = \frac{1326}{4(1/2)(1/2)} = 111 \text{ psi} < 14,400 \text{ psi}$$

CONSIDER ANCHOR BOLTS:

At top anchor:

$$\text{Shear: } \sigma_v = \frac{476}{4} = 116 \text{ lb/bolt}$$

$$\text{Tension: } \sigma_t = \frac{1711}{4} + \frac{3964}{2(17)} = 544 \text{ lb/bolt}$$

these loads appear reasonable for 1/4" concrete anchors.

At bottom anchor:

$$\text{Shear: } \sigma_v = \frac{1326}{4} = 332 \text{ lb/bolt}$$

$$\text{Tension: } \sigma_t = \frac{1269}{4} + \frac{1090}{2(13)} = 359 \text{ lb/bolt}$$

these loads appear reasonable for 3/4" concrete anchors.

CONSIDER WELDS:

Weld to base plate: 3/16" fillet

$$\text{Tension: } \sigma_t = \frac{\frac{1711}{2} - \frac{3964}{4}}{.707(3/16)(4)} = 9432 \text{ psi}$$

D-37

$$\text{shear: } \sigma_v = \frac{1326}{.707(5/16)(2)(23/4)} = 1819 \text{ psi}$$

W4x13 to W4x13: 1/4" fillet

$$\text{shear: } \sigma_v = \frac{1724}{.707(1/4)(2)(23/4)} = 1773 \text{ psi}$$

$$\text{Tension: } \sigma_t = \frac{\frac{1162}{2} + \frac{2724}{4}}{.707(1/4)(4)} = 1789 \text{ psi}$$

@ 1 1/4 x 4 x 6 to W4x13: 5/16" fillet

$$\text{shear: } \sigma_v = \frac{266}{.707(5/16)(3)} = 151 \text{ psi}$$

$$\text{Tension: } \sigma_t = \frac{586 + 789(1/4)}{.707(5/16)(3)} = 689 \text{ psi}$$

Total 840 psi

∴ This support appears acceptable. However, it should be noted that additional loading will be imposed by the other piping supported by this structure. This additional loading will probably not overstress the structure.

Revised loadings of (1482, 496) vs the original loads of (1645, 496) required analysis. Since the revised load decreased the same % as the horizontal load increased, i.e. ~ 3 times the horizontal, and would contribute more to the stress in the bracket element, the support has the same stress order - OK.

SI-71-667-0301

COEFFICIENTS		CONSTRAINTS		RELATIONS	
UNIT	VALUE	UNIT	VALUE	UNIT	VALUE
1	0.000	1	0.000	1	0.000
2	0.000	2	0.000	2	0.000
3	0.000	3	0.000	3	0.000
4	0.000	4	0.000	4	0.000
5	0.000	5	0.000	5	0.000
6	0.000	6	0.000	6	0.000
7	0.000	7	0.000	7	0.000
8	0.000	8	0.000	8	0.000
9	0.000	9	0.000	9	0.000
10	0.000	10	0.000	10	0.000
11	0.000	11	0.000	11	0.000
12	0.000	12	0.000	12	0.000
13	0.000	13	0.000	13	0.000
14	0.000	14	0.000	14	0.000
15	0.000	15	0.000	15	0.000
16	0.000	16	0.000	16	0.000
17	0.000	17	0.000	17	0.000
18	0.000	18	0.000	18	0.000
19	0.000	19	0.000	19	0.000
20	0.000	20	0.000	20	0.000
21	0.000	21	0.000	21	0.000
22	0.000	22	0.000	22	0.000
23	0.000	23	0.000	23	0.000
24	0.000	24	0.000	24	0.000
25	0.000	25	0.000	25	0.000
26	0.000	26	0.000	26	0.000
27	0.000	27	0.000	27	0.000
28	0.000	28	0.000	28	0.000
29	0.000	29	0.000	29	0.000
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33	0.000	33	0.000	33	0.000
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43	0.000	43	0.000	43	0.000
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54	0.000	54	0.000	54	0.000
55	0.000	55	0.000	55	0.000
56	0.000	56	0.000	56	0.000
57	0.000	57	0.000	57	0.000
58	0.000	58	0.000	58	0.000
59	0.000	59	0.000	59	0.000
60	0.000	60	0.000	60	0.000
61	0.000	61	0.000	61	0.000
62	0.000	62	0.000	62	0.000
63	0.000	63	0.000	63	0.000
64	0.000	64	0.000	64	0.000
65	0.000	65	0.000	65	0.000
66	0.000	66	0.000	66	0.000
67	0.000	67	0.000	67	0.000
68	0.000	68	0.000	68	0.000
69	0.000	69	0.000	69	0.000
70	0.000	70	0.000	70	0.000
71	0.000	71	0.000	71	0.000
72	0.000	72	0.000	72	0.000
73	0.000	73	0.000	73	0.000
74	0.000	74	0.000	74	0.000
75	0.000	75	0.000	75	0.000
76	0.000	76	0.000	76	0.000
77	0.000	77	0.000	77	0.000
78	0.000	78	0.000	78	0.000
79	0.000	79	0.000	79	0.000
80	0.000	80	0.000	80	0.000
81	0.000	81	0.000	81	0.000
82	0.000	82	0.000	82	0.000
83	0.000	83	0.000	83	0.000
84	0.000	84	0.000	84	0.000
85	0.000	85	0.000	85	0.000
86	0.000	86	0.000	86	0.000
87	0.000	87	0.000	87	0.000
88	0.000	88	0.000	88	0.000
89	0.000	89	0.000	89	0.000
90	0.000	90	0.000	90	0.000
91	0.000	91	0.000	91	0.000
92	0.000	92	0.000	92	0.000
93	0.000	93	0.000	93	0.000
94	0.000	94	0.000	94	0.000
95	0.000	95	0.000	95	0.000
96	0.000	96	0.000	96	0.000
97	0.000	97	0.000	97	0.000
98	0.000	98	0.000	98	0.000
99	0.000	99	0.000	99	0.000
100	0.000	100	0.000	100	0.000

D-87

[The page contains extremely faint, illegible markings that appear to be bleed-through from the reverse side of the document.]

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SI-01-6007-H306

loads:

Y-dir. $|W + P_d + T| = 626 \text{ lbs.}$

$$|W + P_d + T| + (33E^2 + Am^2)^{1/2} = 705 \text{ lbs.}$$

Lat. - dir. $|W + P_d + T| = 151 \text{ lbs.}$

$$|W + P_d + T| + (33E^2 + Am^2)^{1/2} = 288 \text{ lbs.}$$

This support is essentially the same as SI-01-6007-H30F and the loads applied here are significantly less than those utilized in the analysis of SI-01-6007-H30F. Therefore, this support appears acceptable with the same comments as given for SI-01-6007-H30F.

JZ-01-6007-H30.4

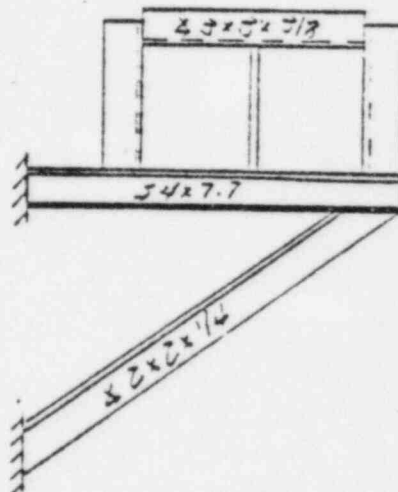
LOADS:

Y-Dir. $|W + P_d + T| = 198 \text{ lb.}$

$|W + P_d + T| + (J^2 E^2 + A m^2)^{1/2} = 347 \text{ lb.}$

Z-Dir. $|W + P_d + T| = 106 \text{ lb.}$

$|W + P_d + T| + (J^2 E^2 + A m^2)^{1/2} = 272 \text{ lb.}$



utilize computer program "FRAME" to find reactions in structural members: Analyze for worst load combination

CONSIDER J4x7.7 MEMBER

$A = 2.26 \text{ in}^2, S = 2.04 \text{ in}^3$

TENSION: $\sigma_t = \frac{551}{2.26} = 244 \text{ psi} < 21,000 \text{ psi}$

Compression: $C_c = 126$

$\frac{L}{r} = \frac{2(13)}{0.581} = 61.96 < 126$

$$F_c = \left[1 - \frac{(61.96)^2}{2(126)^2} \right] 36,000 = 17,235 \text{ psi}$$

Level 1 = 22,402 psi

$\sigma_2 = \frac{552}{2.26} = 245 \text{ psi} < 17,235 \text{ psi}$

Shear: $\tau_v = \frac{222}{2.26} = 97 \text{ psi} < 14,400 \text{ psi}$
D-95

bending: $\sigma_b = \frac{1549}{3.04} = 510 \text{ psi} < 23,760 \text{ psi}$

CONSIDER $\angle 2 \times 2 \times 14$:

TENSION: $\sigma_t = \frac{470}{.938} = 501 \text{ psi} < 21,600 \text{ psi}$

COMPRESSION: $C_c = 126$

$$\frac{L}{r} = \frac{2(25.5)}{.609} = 83.74 < 126$$

$$F_a = \frac{\left[1 - \frac{(83.74)^2}{2(126)^2}\right] 36,000}{\frac{5}{8} + \frac{3(83.74)}{8(126)} - \frac{(83.74)^3}{8(126)^3}} = 14,926 \text{ psi}$$

LEVEL A = 23,061 psi

$$\sigma_a = \frac{466}{.938} = 497 \text{ psi} < 14,926 \text{ psi}$$

shear: $\tau_v = \text{negligible}$

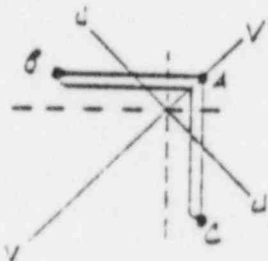
bending:

$$a = .938 \text{ in}^2$$

$$x_o = y_o = 0.592 \text{ in}$$

$$I_x = I_y = 0.343 \text{ in}^4$$

$$r_u = 0.591 \text{ in}$$



$$I_u = a x_o^2 = .938(.591)^2 = 0.143 \text{ in}^4$$

$$I_v = 2(.343) - .143 = 0.553 \text{ in}^4$$

$$v_A = -0.837$$

$$u_A = 0$$

$$v_B = 0.577$$

$$u_B = -1.414$$

$$v_C = 0.577$$

$$u_C = 1.414$$

$$m_u = -.707(18) = -13 \text{ in-lb}$$

$$m_v = .707(18) = 13 \text{ in-lb} \quad D-96$$

$$\sigma_A = \frac{-13(-.777)}{.143} = -76 \text{ psi} < 21,600 \text{ psi}$$

$$\sigma_B = \frac{-13(.577)}{.143} + \frac{13(-1.414)}{.553} = 86 \text{ psi} < 21,600 \text{ psi}$$

$$\sigma_C = \frac{-13(.577)}{.143} + \frac{13(1.414)}{.553} = -42 \text{ psi} < 21,600 \text{ psi}$$

CONSIDER 2nd case:

TENSION: $\sigma_t = \frac{253}{2.11} = 120 \text{ psi} < 21,600 \text{ psi}$

Compression: Negligible

shear: $\sigma_v = \frac{253}{2.11} = 120 \text{ psi} < 14,400 \text{ psi}$

Bending: $a = 2.11 \text{ in}^2$ $I_x = I_y = 1.76$
 $k_a = k_y = 0.888$ $r_u = 0.587$

$$I_u = 2.11(.587)^2 = 0.727 \text{ in}^4$$

$$I_v = 2(1.76) - .727 = 2.793 \text{ in}^4$$

$$v_A = -1.256$$

$$v_B = 0.865$$

$$v_C = .865$$

$$u_A = 0$$

$$u_B = -2.121$$

$$u_C = 2.121$$

$$m_u = -.707(584) = -413 \text{ in-lb}$$

$$m_v = .707(584) = 413 \text{ in-lb}$$

$$\sigma_A = \frac{-413(-1.256)}{.727} = -714 \text{ psi} < 21,600 \text{ psi}$$

$$\sigma_B = \frac{-413(.865)}{.727} + \frac{413(-2.121)}{2.793} = 300 \text{ psi} < 21,600 \text{ psi}$$

$$\sigma_C = \frac{-413(.865)}{.727} + \frac{413(2.121)}{2.793} = -188 \text{ psi} < 21,600 \text{ psi}$$

0-97

CONSIDER # 1/2 x 2 x 7 1/2 :

TENSION: $\sigma_t = \frac{202}{1/2(2)} = 202 \text{ psi} < 21,600 \text{ psi}$

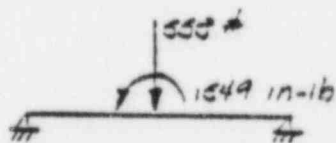
shear: $\sigma_v = \frac{120}{1/2(2)} = 120 \text{ psi} < 14,400 \text{ psi}$

Bending: $\sigma_b = \frac{276(1/2/2)}{2(1/2)^3/12} = 2212 \text{ psi} < 27,000 \text{ psi}$

CONSIDER CASE PLATES:

Analyze plates as if simply supported beams with loads applied at center

Top plate:



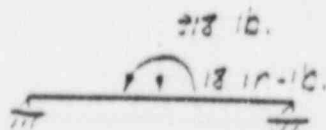
Bending: $\left[1549 + \frac{553(15)}{4} \right] \frac{(3/8/2)}{4(3/8)^3/12} = 28643 \text{ psi}$

$F_b (\text{LEVEL D}) = 1.88(.75)(26,000) = 50,760 \text{ psi} > 28,643 \text{ psi}$

TENSION: $\sigma_t = \frac{176}{[4 - 1 1/4](3/8)} = 180 \text{ psi} < 21,600 \text{ psi}$

shear: $\sigma_v = \frac{176}{2(1 1/2)(3/8)} = 165 \text{ psi} < 14,400 \text{ psi}$

Bottom plate:



D-98

$$\text{Bending: } \left[18 + \frac{313(9)}{4} \right] \frac{(1/2/2)}{6(1/2)^3/12} = 2924 \text{ psi} < 27,000 \text{ psi}$$

$$\text{TENSION: } \sigma_t = \frac{245}{[6-1/2](1/2)} = 125 \text{ psi} < 21,600 \text{ psi}$$

$$\text{Shear: } \sigma_v = \frac{245}{2(1/2)(1/2)} = 230 \text{ psi} < 14,400 \text{ psi}$$

CONSIDER ANCHOR BOLTS:

At top anchor:

$$\text{shear: } \sigma_v = \frac{125}{2} = 93 \text{ lb/bolt}$$

$$\text{TENSION: } \sigma_t = \frac{553}{2} + \frac{1549}{15} = 230 \text{ lb/bolt}$$

These loads appear reasonable for 1" concrete anchors.

$$\text{minimum edge dist} = 15/8 - 1/8 = 1 1/2"$$

At bottom anchor:

$$\text{shear: } \sigma_v = \frac{245}{2} = 173 \text{ lb/bolt}$$

$$\text{TENSION: } \sigma_t = \frac{212}{2} + \frac{17}{5} = 162 \text{ lb/bolt}$$

These loads appear reasonable for 1/2" concrete anchors.

CONSIDER WELDS:

34x7.7 to base plate: 1/4" fillet 0-99

$$\text{TENSION: } \sigma_t = \frac{\frac{553}{2} + \frac{1549}{4}}{.707(1/4)(25/8)} = 1431 \text{ psi}$$

$$\text{shear: } \sigma_v = \frac{136}{.707(1/4)(2)(21/2)} = 210 \text{ psi}$$

4 2x2x1/4 to base plate: 1/4" fillet

$$\text{TENSION: } \sigma_t = \frac{318}{.707(1/4)(8)} = 225 \text{ psi}$$

$$\text{shear: } \sigma_v = \frac{245}{.707(1/4)(8)} = 244 \text{ psi}$$

$$\text{Bending: } \sigma_b = \frac{12}{(.707)(1/4) \left[\frac{4(2)(2) + (2)^2}{6} \right]} = 31 \text{ psi}$$

Total 500 psi

4 3x3x3/8 to 54x7.7: 1/4" fillet

$$\text{TENSION: } \sigma_t = \frac{253}{.707(1/4)(12)} = 119 \text{ psi}$$

$$\text{shear: } \sigma_v = \frac{135}{.707(1/4)(21)} = 87 \text{ psi}$$

$$\text{Bending: } \sigma_b = \frac{574}{(.707)(1/4) \left[\frac{4(2)(2) + (2)^2}{6} \right]} = 441 \text{ psi}$$

D-100

Total: 647 psi

A $1/2 \times 2 \times 7 1/2$ to 34×7.7 :

TENSION: $\sigma_t = \frac{202}{.707(1/4)(5)} = 229 \text{ psi}$

shear: $\sigma_v = \frac{170}{.707(1/4)(5)} = 138 \text{ psi}$

bending: $\sigma_b = \frac{276(1/2)}{.707(1/4)(2)} = 1561 \text{ psi}$

Total: 1929 psi

∴ This support appears acceptable with the most critical stresses occurring in the $3/8"$ base plate. Although a very conservative approach was taken for the plate analysis, it should be considered that loads due to adjacent piping will also be imposed upon this structure.

The loads have been revised:

	Vert.	Horiz.
old load	442	235
new load	347	272
% of old load	79%	116%

The calculation was reviewed, and the baseplate found to control the design. Since a frame analysis was used to run permutations of the possible ± load combinations, results were used to predict the effect of the load change on the baseplate loads and stresses. This was conservatively bounded by multiplying the ratio of increase by the maximum stress ratio:

$$F/F = (1.16)(0.72) = 0.84$$

SI-01-0007-H301

6 D C F D I M A T S									
JOINT NUMBER		X		Y		DISPLACEMENT		RESTRAINTS	

JOINT	FORCES APPLIED IN THE X DIRECTION		CONCENTRATED LOADS IN THE X DIRECTION		APPLIED MOMENT ABOUT THE X AXIS		APPLIED ABOUT THE Y AXIS		APPLIED ABOUT THE Z AXIS		JOINT NUMBER
	X DIRECTION	Y DIRECTION	X DIRECTION	Y DIRECTION	X DIRECTION	Y DIRECTION	X DIRECTION	Y DIRECTION	X DIRECTION	Y DIRECTION	
1	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	1
2	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	2
3	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	3
4	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	4
5	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	5
6	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	6
7	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	7
8	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	8
9	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	9
10	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	10
11	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	11
12	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	12
13	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	13
14	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	14
15	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	15
16	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	16
17	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	17
18	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	18
19	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	19
20	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	20
21	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	21
22	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	22
23	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	23
24	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	24
25	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	25
26	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	26
27	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	27
28	0.0000	0.000									

Q-103

SEISMIC LOGS

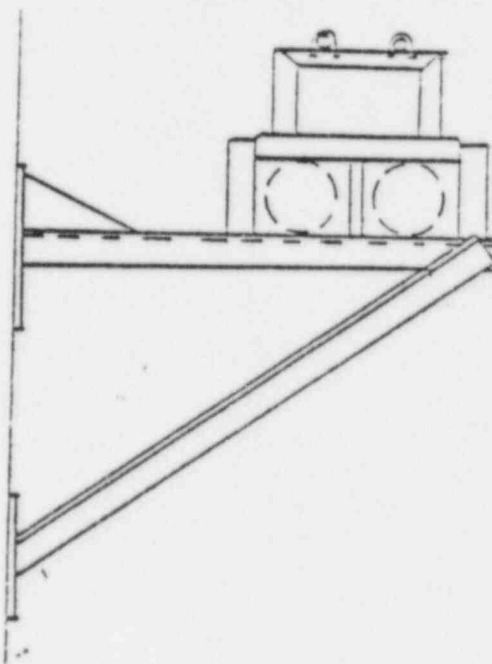
JOINT	MEMBER	AXIAL FORCE	AVERAGE SHEAR FORCE	TORSION	ROTATION	REINFORCING MEMBERS	APPLIED ABOUT THE	APPLIED ABOUT THE	APPLIED ABOUT THE
1	2	3	4	5	6	7	8	9	10
1	1	23200	0	0	0	0	0	0	0
2	2	23200	0	0	0	0	0	0	0
3	3	23200	0	0	0	0	0	0	0
4	4	23200	0	0	0	0	0	0	0
5	5	23200	0	0	0	0	0	0	0
6	6	23200	0	0	0	0	0	0	0
7	7	23200	0	0	0	0	0	0	0
8	8	23200	0	0	0	0	0	0	0
9	9	23200	0	0	0	0	0	0	0
10	10	23200	0	0	0	0	0	0	0
11	11	23200	0	0	0	0	0	0	0
12	12	23200	0	0	0	0	0	0	0
13	13	23200	0	0	0	0	0	0	0
14	14	23200	0	0	0	0	0	0	0
15	15	23200	0	0	0	0	0	0	0
16	16	23200	0	0	0	0	0	0	0
17	17	23200	0	0	0	0	0	0	0
18	18	23200	0	0	0	0	0	0	0
19	19	23200	0	0	0	0	0	0	0
20	20	23200	0	0	0	0	0	0	0
21	21	23200	0	0	0	0	0	0	0
22	22	23200	0	0	0	0	0	0	0
23	23	23200	0	0	0	0	0	0	0
24	24	23200	0	0	0	0	0	0	0
25	25	23200	0	0	0	0	0	0	0
26	26	23200	0	0	0	0	0	0	0
27	27	23200	0	0	0	0	0	0	0
28	28	23200	0	0	0	0	0	0	0
29	29	23200	0	0	0	0	0	0	0
30	30	23200	0	0	0	0	0	0	0
31	31	23200	0	0	0	0	0	0	0
32	32	23200	0	0	0	0	0	0	0
33	33	23200	0	0	0	0	0	0	0
34	34	23200	0	0	0	0	0	0	0
35	35	23200	0	0	0	0	0	0	0
36	36	23200	0	0	0	0	0	0	0
37	37	23200	0	0	0	0	0	0	0
38	38	23200	0	0	0	0	0	0	0
39	39	23200	0	0	0	0	0	0	0
40	40	23200	0	0	0	0	0	0	0
41	41	23200	0	0	0	0	0	0	0
42	42	23200	0	0	0	0	0	0	0
43	43	23200	0	0	0	0	0	0	0
44	44	23200	0	0	0	0	0	0	0
45	45	23200	0	0	0	0	0	0	0
46	46	23200	0	0	0	0	0	0	0
47	47	23200	0	0	0	0	0	0	0
48	48	23200	0	0	0	0	0	0	0
49	49	23200	0	0	0	0	0	0	0
50	50	23200	0	0	0	0	0	0	0
51	51	23200	0	0	0	0	0	0	0
52	52	23200	0	0	0	0	0	0	0
53	53	23200	0	0	0	0	0	0	0
54	54	23200	0	0	0	0	0	0	0
55	55	23200	0	0	0	0	0	0	0
56	56	23200	0	0	0	0	0	0	0
57	57	23200	0	0	0	0	0	0	0
58	58	23200	0	0	0	0	0	0	0
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62	62	23200	0	0	0	0	0	0	0
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66	66	23200	0	0	0	0	0	0	0
67	67	23200	0	0	0	0	0	0	0
68	68	23200	0	0	0	0	0	0	0
69	69	23200	0	0	0	0	0	0	0
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71	71	23200	0	0	0	0	0	0	0
72	72	23200	0	0	0	0	0	0	0
73	73	23200	0	0	0	0	0	0	0
74	74	23200	0	0	0	0	0	0	0
75	75	23200	0	0	0	0	0	0	0
76	76	23200	0	0	0	0	0	0	0
77	77	23200	0	0	0	0	0	0	0
78	78	23200	0	0	0	0	0	0	0
79	79	23200	0	0	0	0	0	0	0
80	80	23200	0	0	0	0	0	0	0
81	81	23200	0	0	0	0	0	0	0
82	82	23200	0	0	0	0	0	0	0
83	83	23200	0	0	0	0	0	0	0
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86	86	23200	0	0	0	0	0	0	0
87	87	23200	0	0	0	0	0	0	0
88	88	23200	0	0	0	0	0	0	0
89	89	23200	0	0	0	0	0	0	0
90	90	23200	0	0	0	0	0	0	0
91	91	23200	0	0	0	0	0	0	0
92	92	23200	0	0	0	0	0	0	0
93	93	23200	0	0	0	0	0	0	0
94	94	23200	0	0	0	0	0	0	0
95	95	23200	0	0	0	0	0	0	0
96	96	23200	0	0	0	0	0	0	0
97	97	23200	0	0	0	0	0	0	0
98	98	23200	0	0	0	0	0	0	0
99	99	23200	0	0	0	0	0	0	0
100	100	23200	0	0	0	0	0	0	0

0-104

[illegible]

(continued from page 9)

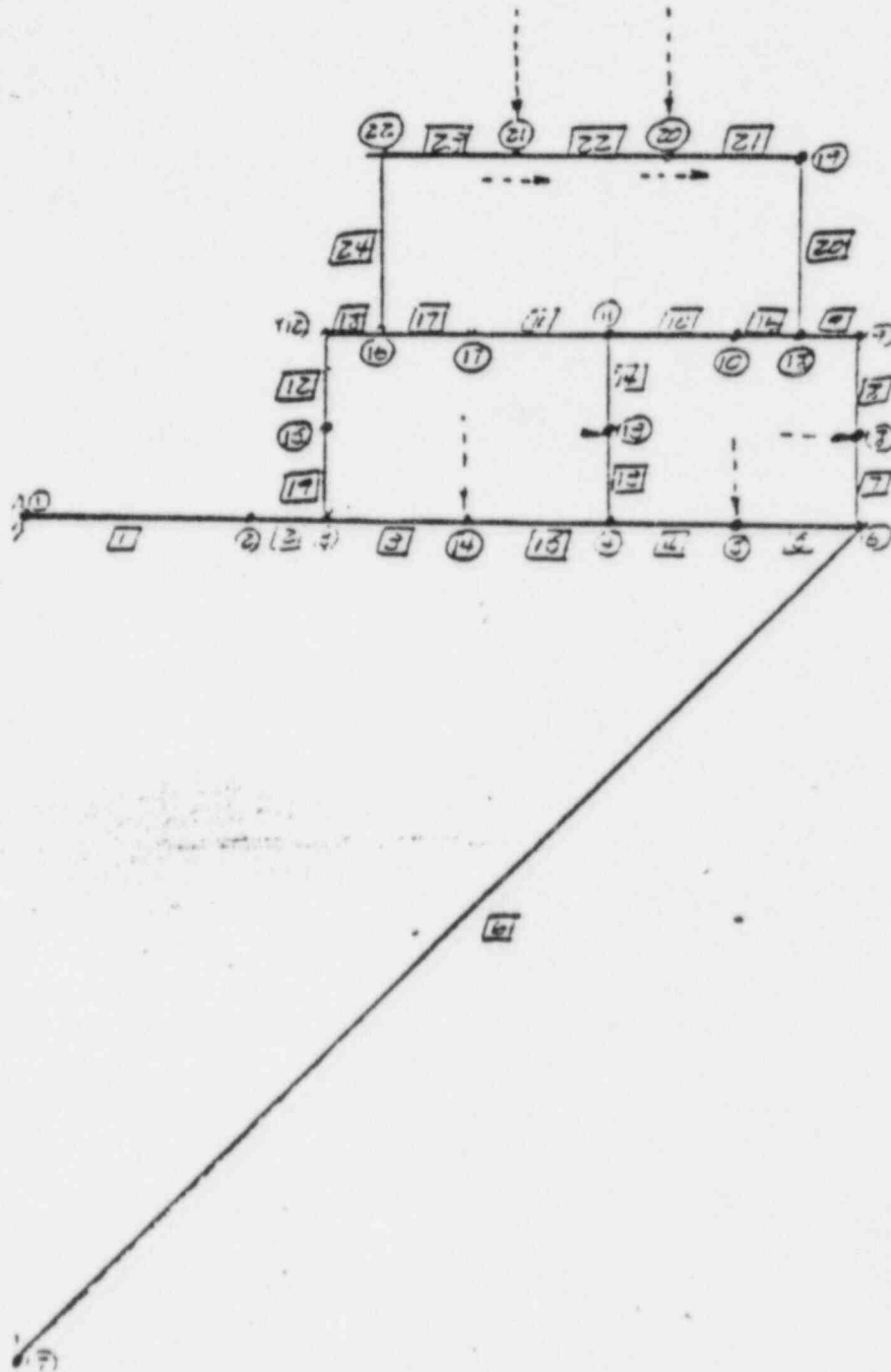
SI-01-6007-H30I



Load equation $|W + Pd + T| + (SSE^2 + AM^2)^{1/2}$

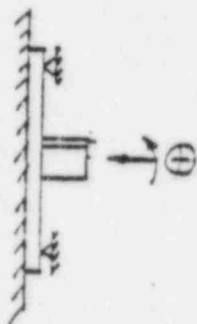
Load Case	Load Description	Node	X-Force lb	Y-Force lb	Z-Force lb
1	-X	5	638.0	0	0
	+X	13	915.0	0	0
	+X	20	100	0	0
	+X	21	100	0	0
2	-X	13	-656.0	0	0
	-X	19	-1460.0	0	0
	-X	20	-100	0	0
	-X	21	-100	0	0
3	-Y	1	0	705.0	0
	-Y	13	0	0.00	0
	-Y	20	0	100.0	0
	-Y	21	0	100.0	0
4	-Y	12	0	-933.0	0
	-Y	19	0	-279.0	0
	-Y	20	0	-20.0	0
	-Y	21	0	-20.0	0

Use computer program "Frame" to find member structure members:



Consider Case Plates

Analyze plates as if simply supported beams with displacements allowed only in positive X-direction and loads applied at center.



Top plate $A = 4(\frac{3}{8}) = 1.50 \text{ in}^2$

$S = \frac{1}{2}(4)(\frac{3}{8})^3 / (\frac{3}{4}) = 0.094 \text{ in}^3$

$L = 14$

Bottom plate $A = 6(\frac{1}{2}) = 3.00 \text{ in}^2$

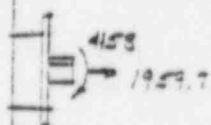
$S = \frac{1}{2}(6)(\frac{1}{2})^3 / (\frac{1}{4}) = 0.250 \text{ in}^3$

$L = 6"$

Consider bending in top plate

Load Case	X-Reaction lbs	Y-Reaction lbs	Moment React. lb-in	Load Type
1	-2251	-532.7	-1425	+X
2	3352	688.2	587.8	-X
3	291.3	-585.4	-2733	-Y
4	-550.9	732.3	3871	-Y

Combine Load case 1 & 3

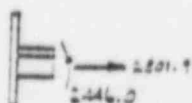


$X\text{-React} = -2251 - 291.3 = -1959.7$

$\text{Moment React} = -1425 - 2733 = -4158$

$T_b = \left[\frac{1959.7}{4} (14) + 4158 \right] \frac{1}{0.094} = 117,201.6 \text{ psi}$

Combine Load cases 1 & 4



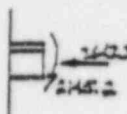
$X\text{-React} = -2251 - 550.9 = -2801.9 \text{ lbs}$

$\text{Moment} = -1425 + 3871 = 2446.0 \text{ lb-in}$

$T_b = \left[\frac{2801.9}{4} (14) - 2446 \right] \frac{1}{0.094} = 130,347.3 \text{ psi}$

Combine Load cases 2 & 3

$$R - \text{React} = 3352 + 291.3 = 3643.3$$

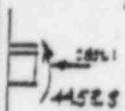


$$\text{Moment React} = 587.8 - 2733 = -2145.2$$

$$T_b = \frac{2145.2}{0.094} = 22,821.3$$

Combine load cases 2 & 4

$$R - \text{React} = 3352 - 550.9 = 2801.1$$



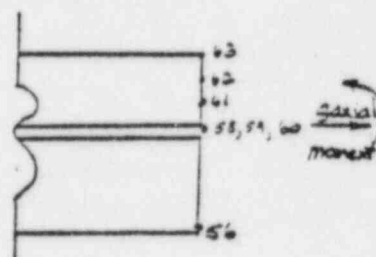
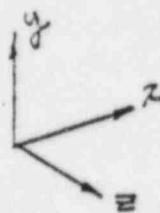
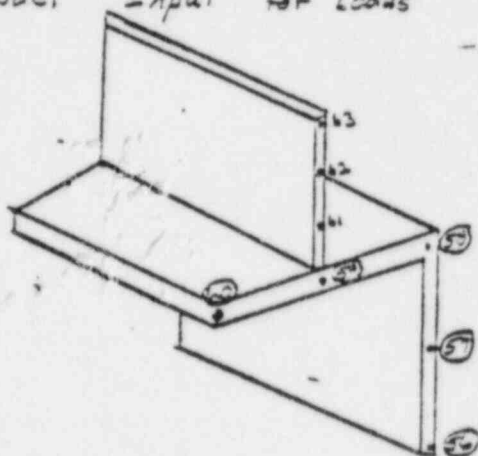
$$\text{Moment React} = 587.8 + 3971 = 4458.8$$

$$T_b = \frac{4458.8}{0.094} = 47,434.2 \text{ psi}$$

The combination of load cases 1 and 3 and the combination of load cases 1 and 4 give the largest bending stresses in the plate. Idealizing the base plate as a simply supported beam gave very high stresses so a finite element model and SUPERSAP were used to determine more realistic stresses.

The stresses for the plate were determined for two load cases. Load case 1 is the combination of cases 1 & 3 and load case 2 is the combination of cases 1 & 4 above.

Super Sap Model Input for Loads



Load Case 1

$$\text{Axial load} = 2251 - 291.3 = 1959.7 \text{ lbs}$$

$$\text{Moment} = -1425 - 2733 = -4158 \text{ lb-in}$$

$$\frac{1959.7}{4} = 489.9 \text{ lbs} \quad \frac{4158}{2} = 2079 \text{ lbs}$$

$$\frac{-4158 \text{ lb-in}}{3 \text{ lb}} = 1386 \text{ lbs} \dots$$

Node	X-Force	Y-Force	Z-Force
56	0	0	1631.0
57	0	0	489.1
58	0	0	-896.1
59	0	0	489.9
60	0	0	245

$$56 \Rightarrow 245.0 + 1386 = 1631.0$$

$$57 \Rightarrow 489.9$$

$$58 \Rightarrow 489.9 - 1386 = -896.1$$

$$59 \Rightarrow 489.9$$

$$60 \Rightarrow 245$$

D-111

Load Case 2

$$\text{Axial load} = 2251 + 550.9 = 2801.9$$

$$\text{Moment} = -1425 + 3871 = 2446.0$$

$$\frac{2801.9}{4} = 700.5 \quad \frac{2446.0}{2} = 1223.0$$

$$\frac{2446.0}{3} \frac{\text{lb-in}}{\text{in}} = 815.3 \text{ lbs (couple force)}$$

Node	X-Force	Y-Force	Z-Force
56	0	0	1165.5
57	0	0	700.5
58	0	0	-1148
59	0	0	700.5
60	0	0	350.2

Node

$$56 \quad 350.2 + 815.3 = 1165.5$$

$$57 \quad 700.5 =$$

$$58 \quad 700.5 - 815.3 = -114.8$$

$$59 \quad 700.5$$

$$60 \quad 350.2$$

D-112

Loading the SUPERSAP model with the loads given on pages 3 and 4 give plate bending stresses to be approximately 35.5 KSI in the area just below the angle.

$$\text{Bending stress} = 35.5 \text{ KSI} \leq \text{allowable} = 50.76 \text{ KSI} = 0.75(36)(1.08) \text{ for level D.}$$

The upper base plate is subjected to larger loads than the lower base plate; therefore the lower base plate will not be checked for bending.

Check Anchor Bolts in lower base plate

$$X = 291.3 + 635.9 = 927.2$$

$$Y = 319.6 + 689.2 = 1007.8 \text{ lbs}$$



$$M = 353.1 + 127.1 = 480.2 \text{ lb-in}$$

$$\text{Maximum tension} = \frac{927.2}{2} \div \frac{190.2}{6} = 543.6 \text{ lbs/bolt}$$

$$\text{Maximum shear} = \frac{1007.8}{2} = 503.9 \text{ lbs/bolt}$$

Check Anchor Bolts in upper base plate

$$\text{Maximum tension (from super sap)} = 261.7 + 1367.0 + 731.2 + 1957.5 + 247.7 = 3972 \text{ lbs}$$

$$\text{Maximum shear (from frame)} = \frac{732.3}{2} + \frac{689.2}{2} = 710.3 \text{ lbs/bolt}$$

Check $23 \times 3 \times \frac{1}{2}$

Tension:

(BX + BY member 1)

$$\text{Axial Force} = 550.9 + 2251 = 2801.9$$

$$f_t = \frac{2801.9}{2.75} = 1019 \text{ psi} \leq 21,000 = 0.6(36)$$

Compression:

$$C_a = 126.1 \text{ p.s.f. AISI}$$

$$\frac{KL}{r} = \frac{(2.0)(12)}{0.564} = 61.64 < 126.1$$

$$F_a = 67,270 \text{ p.s.f. AISI} (1.88 \text{ level D}) = 32,470 \text{ psi}$$

(BX + BY member 1)

$$\text{Axial Force} = 3352 + 291.3 = 3643 \text{ lbs}$$

$$f_a = \frac{3643}{2.75} = 1325 \text{ psi}$$

$$f_a = 1325 \text{ psi} \leq F_a = 32,470$$

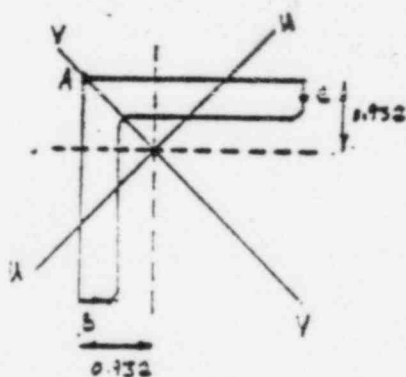
Bending:

$$\text{Area} = 2.75$$

$$x_o = y_o = 0.932$$

$$I_x = I_y = 2.33$$

$$r_x = 0.594$$



$$I_u = A r_o^2 = 2.75 (0.594)^2 = 0.939 \text{ in}^4$$

$$I_u + I_v = I_x + I_y$$

$$I_v = 2(2.33) - 0.939 = 3.502 \text{ in}^4$$

$$y_o = -1.318 \quad x_o = 0.803 \quad y_u = 0.803$$

$$u_o = 0 \quad u_u = -2.121 \quad u_v = 2.121$$

$$M_u = -707 (4459) = -3152$$

$$M_v = 0.707 (4459) = 3152$$

$$T_1 = \frac{M_u u_o}{I_u} - \frac{M_v u_v}{I_v} = \frac{-3152(-2.121)}{0.939} = 4461 \text{ psi}$$

$$T_2 = \frac{-3152(0.803)}{0.939} - \frac{3152(-2.121)}{3.502} = -4607 \text{ psi}$$

$$T_3 = \frac{-3152(0.803)}{0.939} - \frac{3152(2.121)}{3.502} = -4607 \text{ psi}$$

$$f_{b, \text{max}} = 4607 \text{ psi} \leq f_{b, \text{min}} = 0.6(36) = 21,000 \text{ psi}$$

D-114

$$\begin{aligned} \text{Moment} &= 707 \times 2 \times 1.752 \\ &= 2471 \text{ lb-in} \\ &= 4459 \text{ lb-in} \end{aligned}$$

Shear:

$$\text{Force} = 683.2 + 732 = 1415.2 \text{ lbs}$$
$$\text{SX} + \text{SY}$$

$$\text{shear area} = (3)(1/2) = 1.5 \text{ in}^2$$

$$f_v = \frac{1415.2 \text{ lbs}}{1.5 \text{ in}^2} = 943 \text{ psi} \leq (0.4)(36)(1.89) = 27,072 \text{ psi}$$

$$f_v = 943 \text{ psi} \leq F_v = 27,072 \text{ psi}$$

Check $\lambda = r_2 \times 3/8$

Tension:

$$\text{Force} = (\text{SX} + \text{SY member 6})$$

$$\text{Force} = 937 + 432 = 1369 \text{ lbs}$$

$$f_t = \frac{1369}{1.36} = 1007 \text{ lbs} \leq F_t = 40,600 \text{ lbs}$$

Compression:

$$\text{Force} = (\text{SX} + \text{SY member 6})$$

$$\text{Force} = 729 + 819 = 1548 \text{ lbs}$$

$$f_c = \frac{1548}{1.36} = 1138 \text{ psi}$$

$$C_c = 126 \text{ p.s.i. - Allow}$$

$$\frac{\lambda}{r} = \frac{2.2(31)}{0.389} = 157.4 > 126$$

$$F_c = 5,978 (1.99) = 11,051 \text{ psi}$$

p.s.i. Allow

$$f_c = 1138 \leq F_c = 11,051 \text{ psi}$$

0-115

Bending:

$$\text{Moment} = (\oplus X + \oplus Y \text{ member})$$

$$\text{Moment} = 161.6 + 34.7 = 196.3$$

$$A = 1.36$$

$$I_x = I_y = 0.479$$

$$x_o = y_o = 0.636$$

$$r_u = 0.389$$

$$I_u = 1.36(0.389)^2 = 0.206$$

$$I_v = 2(0.479) - 0.206 = 0.752$$

$$v_A = -0.849 \quad v_B = 0.515 \quad v_C = 0.515$$

$$u_A = 0.0 \quad u_B = -1.44 \quad u_C = 1.414$$

$$m_u = -0.707(196.3) = -138.9 \text{ lb-in}$$

$$m_v = 0.707(196.3) = 138.9 \text{ lb-in}$$

$$T_A = \frac{-138.9(-0.849)}{0.206} = 565.7 \text{ psi}$$

$$T_B = \frac{-138.9(0.515)}{0.206} + \frac{138.9(-1.414)}{0.752} = -609 \text{ psi}$$

$$T_C = \frac{-138.9(0.515)}{0.206} + \frac{138.9(1.414)}{0.752} = -86 \text{ psi}$$

$$F_{\text{min}} = 0.6(36)(1.92) = 40,600$$

$$F_{\text{max}} = 609 \text{ psi} \leq F_{\text{min}} = 40,600$$

Shear:

$$\text{Force} = (\oplus X + \oplus Y \text{ member})$$

$$\text{Force} = 5.9 + 2.1 = 7.9 \text{ lbs}$$

$$\text{shear area} = 2(2.5) = 5.0 \text{ in}^2$$

$$F_v = \frac{7.9}{5.0} = 1.58 \text{ psi} \leq 0.4(26,000) = 10,400$$

$$F_v = 1.58 \leq F_v = 27,000 \text{ psi}$$

Check $\angle 3 \times 3 \times 7/8$

Tension:

Force = 92 + 94 member 12

$$\text{Force} = 629 + 691 = 1320 \text{ lbs}$$

$$F_t = \frac{1320}{2.11} = 621 \text{ psi} \quad F_t = 0.6(36,000)(1.184) = 40,320$$

$$F_t = 621 \text{ psi} \leq F_t = 40,320 \text{ psi}$$

Compression:

$$C_u = 12.6 \text{ p. 5-16 AISC}$$

$$\frac{L}{r} = \frac{2.2(12)}{0.537} = 40.2 < 12.6 \text{ [member which is braced} \\ \text{crossed end of} \\ \text{member - 12]}$$

$$F_c = 19,120 \text{ psi (19.12)} = 35,440$$

Force = (92 + 94 members 12 + 1)

$$\text{Force} = 662 + 522 = 1,184 \text{ lbs}$$

$$F_c = \frac{1,184}{2.11} = 561 \text{ psi}$$

$$F_c = 561 \leq F_c = 35,440$$

Shear:

Force = 92 + 94 member 12

$$\text{Force} = 1966 + 209 = 2,174 \text{ lbs}$$

$$\text{Shear} - \text{area} = (3)(7/8) = 1.25 \text{ in}^2$$

$$F_v = \frac{2,174 \text{ lbs}}{1.25 \text{ in}^2} = 1,739 \text{ psi}$$

$$F_v = 0.4(36,000)(1.184) = 27,000 \text{ psi}$$

$$F_v = 1,739 \text{ psi} \leq F_v = 27,000 \text{ psi}$$

Bending:

$$\text{Moment} = 3X + 3Y \text{ girders } \times \text{moment} = 3$$

$$\text{Moment} = 3049 + 451 = 3500 \text{ lb-ft}$$

$$A = 211 \text{ in}^2 \quad I_x = I_y = 1.76 \text{ in}^4$$

$$Z_x = Y_c = 0.555 \quad r_x = 0.587 \text{ in}$$

$$I_u = 211 (0.587)^2 = 0.727 \text{ in}^4$$

$$I_v = 2 (1.76 - 0.727) = 2.793 \text{ in}^4$$

$$u_x = -1.256 \quad u_y = 0.725 \quad u_z = 0.725$$

$$u_x = 0.1 \quad u_y = -2.121 \quad u_z = 2.121$$

$$M_x = -0.725 (3500) = -2538$$

$$M_y = 0.725 (3500) = 2538$$

$$T_1 = \frac{-2538 (-1.256)}{0.727} = 3608$$

$$T_2 = \frac{-2538 (0.725)}{0.727} + \frac{2538 (2.121)}{2.793} = -4043$$

$$T_3 = \frac{-2538 (0.725)}{0.727} + \frac{2538 (2.121)}{2.793} = -404$$

$$T_2 \text{ maximum} = 4043 \leq F_{\text{minimum}} = 40,500$$

Check plate $1/2 \times 2 \times 7/2$

Tension: Force = 395 lbs

$$F_t = \frac{395}{2(1)} = 395 \text{ psi}$$

$$F_t = 395 \text{ psi} \leq F_t = 40,500 \text{ psi}$$

Compression:

Force = 358 lbs $F_c = \frac{358}{2(1)} = 358 \text{ psi}$

$$I = \frac{1}{12} (2) (1)^3 = 0.167 \quad \lambda = \sqrt{\frac{I}{A}} = \sqrt{\frac{0.167}{2}} = 0.289$$

0-118

$$F_t = \frac{(2)(7.5)}{0.1243} = 103.9 \quad C_u = 1.26$$

$$F_u = (12,482)(1.26) = 15,726$$

$$F_u = 359 \leq 15,726 = F_u$$

Bending:

$$\text{Moment} = (92 + 94) \text{ kips} \times \text{member 3}$$

$$\text{Moment} = 1354 + 1575 = 2929 \text{ ft-kips}$$

$$F_b = \frac{(2929)(12)}{0.0239} = 148,240$$

$$F_b = (0.75)(36,000)(1.39) = 37,760$$

$$F_b = 148,240 \leq F_b = 37,760$$

Check Welds

Welds are at least $\frac{1}{4}$ " fillet weld and are all around unless noted otherwise.

$$\text{Weld Capacity per inch of weld} = (0.707)(\frac{1}{4})(148,240 \text{ p.s.i.}) = \underline{10,371 \text{ lbs/in}}$$

$$\text{Connection of } 2 \times 2 \times \frac{7}{8} \Rightarrow \text{length} = 2[2 + 2 + \frac{7}{8} - \frac{7}{8}] = 8.5"$$

$$3 \times 3 \times \frac{7}{8} \Rightarrow \text{length} = 2[3 + 3 - \frac{7}{8} - \frac{7}{8}] = 13.5"$$

$$3 \times 2 \times \frac{1}{2} \Rightarrow \text{length} = 2[3 + 3 - \frac{1}{2} - \frac{1}{2}] = 14.0"$$

$$\text{Min. min weld strength} = 8.5(10,371 \text{ lbs/in}) = 88,154 \text{ lbs}$$

$$\text{Min. min weld strength} = 40,950 \text{ lbs} \Rightarrow \text{to any member group.}$$

FRAME OUTPUT

Pages 14 thru 29

0-120

14251.04
4981.00
2321.00
2221.00

ROTATION ABOUT THE Z AXIS
DISPLACEMENT IN THE X DIRECTION
DISPLACEMENT IN THE Y DIRECTION
ROTATION ABOUT THE X AXIS

D-123

[illegible]

The diagram illustrates a coordinate system for navigation or surveying. It features two primary axes:

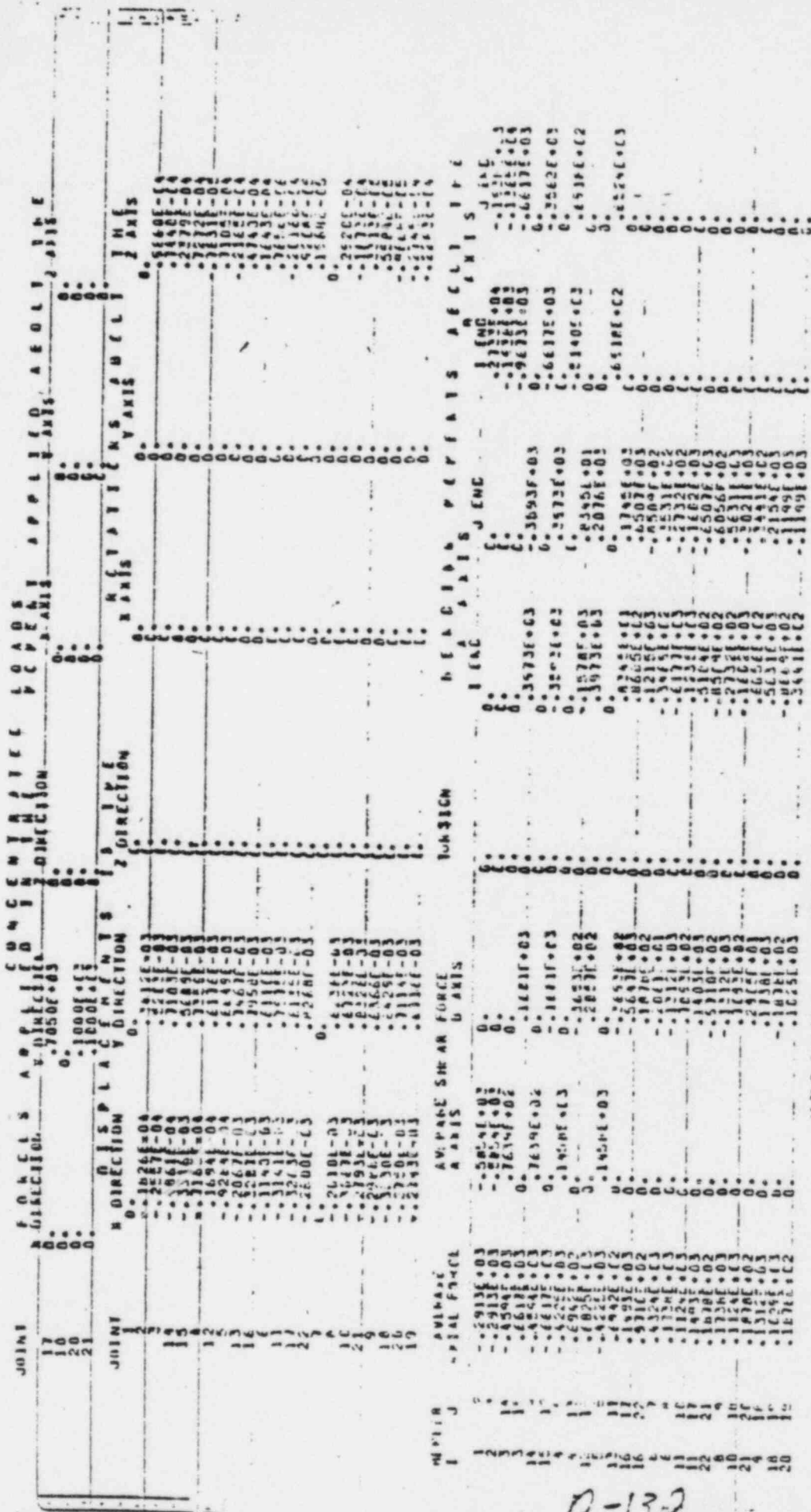
- Z AXIS**: Vertical axis pointing upwards.
- X AXIS**: Horizontal axis pointing to the right.
- Y AXIS**: Vertical axis pointing downwards (labeled as such).
- X AXIS**: Horizontal axis pointing to the left.

A dashed line connects the origin of the X-Z plane to the origin of the X-Y plane. Various labels are present around the diagram:

- DIRECTION**: Located near the top right.
- ELEVATION**: Located near the middle right.
- AZIMUTH**: Located near the bottom right.
- DISTANCE**: Located near the bottom right.

[illegible][illegible]

POSITIVE Y LEADING



D-132

2351.04
2412.03
2408.02
2351.02

ROTATION AB(U) 112 2 215
DISPLACEMENT IN THE X DIRECTION
DISPLACEMENT IN THE Y DIRECTION
ROTATION ABOUT THE Z AXIS

1971E-04
1971E-03
1971E-02
1971E-01

ROTATION ABOUT THE Z AXIS
DISPLACEMENT IN THE X-DIRECTION
DISPLACEMENT IN THE Y-DIRECTION
DISPLACEMENT IN THE Z-DIRECTION

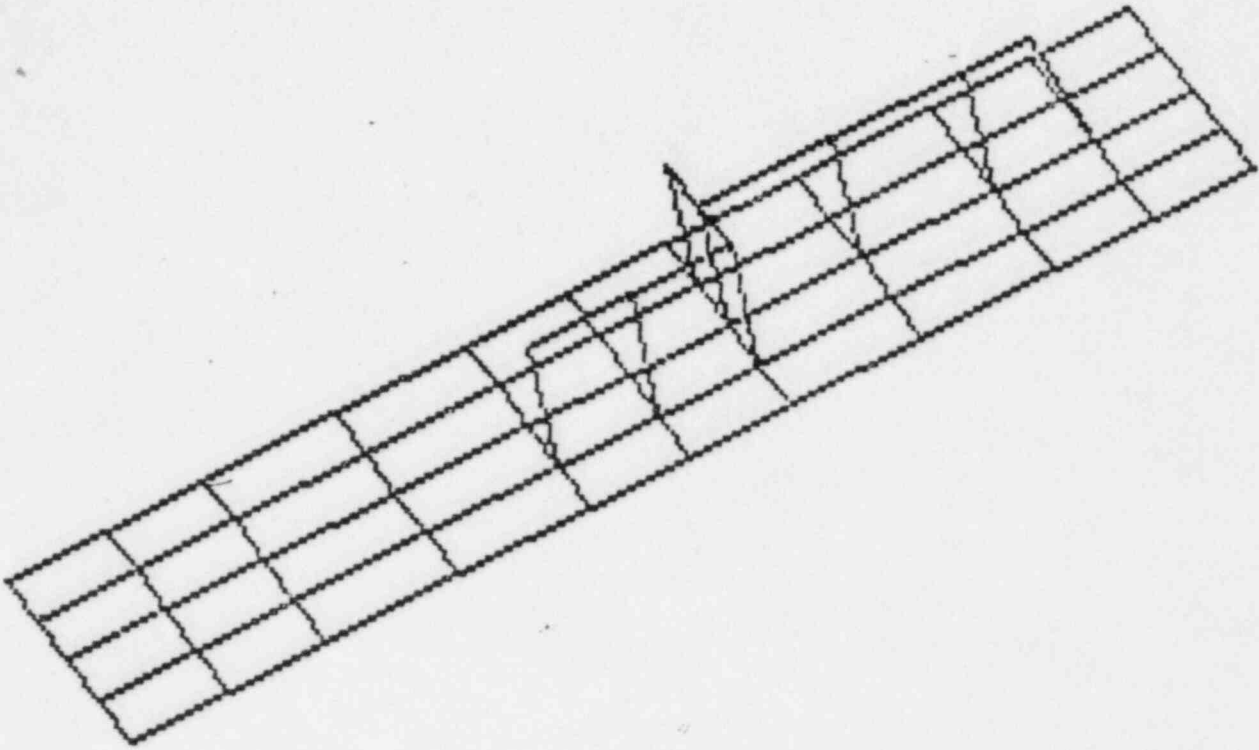
D-135

SUPERSAP OUTPUT

Pages 31 thru 42

D-137

Z-AXIS



Y-AXIS

D-138

Write fault error writing device PRN
Abort. Retry. Ignore? A

CC:\>TYPE CLINT1.L

1**** SUPERSAP ANALYSIS VERSION:
RELEASED:

7.1
7/30/85

DATE: June 11, 1986

TIME: 08:08:30

INPUT FILE.....CLINT1

BASE PLATE WITH GUSSET

1**** CONTROL INFORMATION

NUMBER OF NODE POINTS	(NUMNP)	=	63
NUMBER OF ELEMENT TYPES	(NELTYP)	=	2
NUMBER OF LOAD CASES	(LL)	=	2
NUMBER OF FREQUENCIES	(NF)	=	0
GEOMETRIC STIFFNESS FLAG	(GEOSTF)	=	0
ANALYSIS CODE	(NDYN)	=	0
SOLUTION MODE	(MODEX)	=	0
NUMBER OF ITERATION VECTORS	(NAD)	=	0
EQUATIONS PER BLOCK	(KEOB)	=	0
TAPE10 SAVE FLAG	(NIOSV)	=	0
NODAL DEFLECTION PRINTING	(DEFPCH)	=	0
OVERALL MATRIX PRINTING	(GENPRT)	=	0
ELEMENT MATRIX PRINTING	(ELPRT)	=	0
WEIGHT AND C.G. FLAG	(IWTCG)	=	0
BANDWIDTH MINIMIZATION FLAG	(MINBND)	=	0
FILE FORMAT FLAG	(IPLT)	=	0
NUMBER OF RESPONSE SPECTRA	(NRSC)	=	0
GRAVITATIONAL CONSTANT	(GRAV)	=	3.8640E+02
TOTAL BLANK COMMON	(MTOT)	=	23000

SANDWIDTH MINIMIZATION SPECIFIED

1**** NODAL DATA

NODE NO.	BOUNDARY CONDITION CODES						NODAL POINT COORDINATES			
	DX	DY	DZ	FX	FY	FZ	X	Y	Z	T
1	0	1	1	0	0	0	1.000E+00	1.000E+00	1.000E+00	1.000E+00
2	0	0	0	0	0	0	1.000E+00	1.000E+00	1.000E+00	1.000E+00
3	0	0	0	0	0	0	2.000E+00	1.000E+00	1.000E+00	1.000E+00
4	0	0	0	0	0	0	3.000E+00	1.000E+00	1.000E+00	1.000E+00
5	0	1	1	0	0	0	4.000E+00	1.000E+00	1.000E+00	1.000E+00
6	0	0	0	0	0	0	1.000E+00	1.500E+00	1.000E+00	1.000E+00
7	0	0	0	0	0	0	1.000E+00	1.500E+00	1.000E+00	1.000E+00
8	0	0	0	0	0	0	2.000E+00	1.500E+00	1.000E+00	1.000E+00
9	0	0	0	0	0	0	3.000E+00	1.500E+00	1.000E+00	1.000E+00
10	0	0	0	0	0	0	4.000E+00	1.500E+00	1.000E+00	1.000E+00
11	0	0	0	0	0	0	1.000E+00	3.000E+00	1.000E+00	1.000E+00
12	0	0	0	0	0	0	1.000E+00	3.000E+00	1.000E+00	1.000E+00
13	0	0	0	0	0	0	2.000E+00	3.000E+00	1.000E+00	1.000E+00
14	0	0	0	0	0	0	3.000E+00	3.000E+00	1.000E+00	1.000E+00
15	0	0	0	0	0	0	4.000E+00	3.000E+00	1.000E+00	1.000E+00
16	0	0	0	0	0	0	1.000E+00	5.000E+00	1.000E+00	1.000E+00
17	0	0	0	0	0	0	1.000E+00	5.000E+00	1.000E+00	1.000E+00
18	0	0	0	0	0	0	2.000E+00	5.000E+00	1.000E+00	1.000E+00
19	0	0	0	0	0	0	3.000E+00	5.000E+00	1.000E+00	1.000E+00

21	0	0	0	0	0	0	7.000E+00	7.000E+00	.000E+00	.000E+00
22	0	0	0	0	0	0	.000E+00	7.000E+00	.000E+00	.000E+00
23	0	0	0	0	0	0	1.000E+00	7.000E+00	.000E+00	.000E+00
24	0	0	0	0	0	0	2.000E+00	7.000E+00	.000E+00	.000E+00
25	0	0	0	0	0	0	3.000E+00	7.000E+00	.000E+00	.000E+00
26	0	0	0	0	0	0	4.000E+00	7.000E+00	.000E+00	.000E+00
27	0	0	0	0	0	0	.000E+00	8.500E+00	.000E+00	.000E+00
28	0	0	0	0	0	0	1.000E+00	8.500E+00	.000E+00	.000E+00
29	0	0	0	0	0	0	2.000E+00	8.500E+00	.000E+00	.000E+00
30	0	0	0	0	0	0	3.000E+00	8.500E+00	.000E+00	.000E+00
31	0	0	0	0	0	0	4.000E+00	8.500E+00	.000E+00	.000E+00
32	0	0	0	0	0	0	.000E+00	1.000E+01	.000E+00	.000E+00
33	0	0	0	0	0	0	1.000E+00	1.000E+01	.000E+00	.000E+00
34	0	0	0	0	0	0	2.000E+00	1.000E+01	.000E+00	.000E+00
35	0	0	0	0	0	0	3.000E+00	1.000E+01	.000E+00	.000E+00
36	0	0	0	0	0	0	4.000E+00	1.000E+01	.000E+00	.000E+00
37	0	0	0	0	0	0	.000E+00	1.200E+01	.000E+00	.000E+00
38	0	0	0	0	0	0	1.000E+00	1.200E+01	.000E+00	.000E+00
39	0	0	0	0	0	0	2.000E+00	1.200E+01	.000E+00	.000E+00
40	0	0	0	0	0	0	3.000E+00	1.200E+01	.000E+00	.000E+00
41	0	0	0	0	0	0	4.000E+00	1.200E+01	.000E+00	.000E+00
42	0	0	0	0	0	0	.000E+00	1.400E+01	.000E+00	.000E+00
43	0	0	0	0	0	0	1.000E+00	1.400E+01	.000E+00	.000E+00
44	0	0	0	0	0	0	2.000E+00	1.400E+01	.000E+00	.000E+00
45	0	0	0	0	0	0	3.000E+00	1.400E+01	.000E+00	.000E+00
46	0	0	0	0	0	0	4.000E+00	1.400E+01	.000E+00	.000E+00
47	0	0	0	0	0	0	.000E+00	1.550E+01	.000E+00	.000E+00
48	0	0	0	0	0	0	1.000E+00	1.550E+01	.000E+00	.000E+00
49	0	0	0	0	0	0	2.000E+00	1.550E+01	.000E+00	.000E+00
50	0	0	0	0	0	0	3.000E+00	1.550E+01	.000E+00	.000E+00
51	0	0	0	0	0	0	4.000E+00	1.550E+01	.000E+00	.000E+00
52	0	1	1	0	0	0	.000E+00	1.700E+01	.000E+00	.000E+00
53	0	0	0	0	0	0	1.000E+00	1.700E+01	.000E+00	.000E+00
54	0	0	0	0	0	0	2.000E+00	1.700E+01	.000E+00	.000E+00
55	0	0	0	0	0	0	3.000E+00	1.700E+01	.000E+00	.000E+00
56	0	1	1	0	0	0	4.000E+00	1.700E+01	.000E+00	.000E+00
57	0	0	0	0	0	0	3.000E+00	7.000E+00	2.000E+00	.000E+00
58	0	0	0	0	0	0	3.000E+00	8.500E+00	2.000E+00	.000E+00
59	0	0	0	0	0	0	3.000E+00	1.000E+01	2.000E+00	.000E+00
60	0	0	0	0	0	0	2.000E+00	1.000E+01	2.000E+00	.000E+00
61	0	0	0	0	0	0	1.000E+00	1.000E+01	2.000E+00	.000E+00
62	0	0	0	0	0	0	2.000E+00	1.200E+01	2.000E+00	.000E+00
63	0	0	0	0	0	0	2.000E+00	1.400E+01	2.000E+00	.000E+00
							2.000E+00	1.500E+01	2.000E+00	.000E+00

1**** THIN PLATE/SHELL ELEMENTS

NUMBER OF ELEMENTS = 47
NUMBER OF MATERIALS = 1

1**** MATERIAL PROPERTIES

MATERIAL I.D. NUMBER = 1
WEIGHT DENSITY = 2.5300E-01
MASS DENSITY = 7.199E-04

CHV/ ALPHA(1)	CHV/ ALPHA(2)	CAS	CHV	CAS	CHV
3.187E+07	5.550E+00	.000E+00	3.187E+07	.000E+00	1.115E+07
.000E+00	.000E+00				

1**** ELEMENT LOAD MULTIPLIERS

CASE A	CASE B	CASE C	CASE D
			0-140

TEMP	.000E+00	.000E+00	.000E+00	.000E+00
X-DIR	.000E+00	.000E+00	.000E+00	.000E+00
Y-DIR	.000E+00	.000E+00	.000E+00	.000E+00
Z-DIR	.000E+00	.000E+00	.000E+00	.000E+00

***** ELEMENT CONNECTIVITY DATA

ELEM NO.	NODE I	NODE J	NODE K	NODE L	NODE O	MAT'L INDEX	AVERAGE THICKNESS	NORMAL PRESSURE	TEMP. DIFFERENCE	TEMP. GRADIENT
1	1	2	7	6	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
2	6	7	12	11	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
3	11	12	17	16	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
4	16	17	22	21	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
5	21	22	27	26	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
6	26	27	32	31	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
7	31	32	37	36	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
8	36	37	42	41	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
9	41	42	47	46	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
10	46	47	52	51	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
11	2	3	8	7	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
12	7	8	13	12	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
13	12	13	18	17	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
14	17	18	23	22	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
15	22	23	28	27	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
16	27	28	33	32	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
17	32	33	38	37	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
18	37	38	43	42	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
19	42	43	48	47	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
20	47	48	53	52	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
21	3	4	9	8	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
22	8	9	14	13	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
23	13	14	19	18	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
24	18	19	24	23	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
25	23	24	29	28	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
26	28	29	34	33	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
27	33	34	39	38	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
28	38	39	44	43	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
29	43	44	49	48	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
30	48	49	54	53	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
31	4	5	10	9	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
32	9	10	15	14	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
33	14	15	20	19	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
34	19	20	25	24	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
35	24	25	30	29	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
36	29	30	35	34	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
37	34	35	40	39	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
38	39	40	45	44	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
39	44	45	50	49	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
40	49	50	55	54	0	1	3.750E-01	.000E+00	.000E+00	.000E+00
41	24	29	37	36	0	1	5.000E-01	.000E+00	.000E+00	.000E+00
42	29	34	38	37	0	1	5.000E-01	.000E+00	.000E+00	.000E+00
43	34	39	39	38	0	1	5.000E-01	.000E+00	.000E+00	.000E+00
44	39	44	40	39	0	1	5.000E-01	.000E+00	.000E+00	.000E+00
45	44	49	41	40	0	1	5.000E-01	.000E+00	.000E+00	.000E+00
46	49	54	42	41	0	1	5.000E-01	.000E+00	.000E+00	.000E+00
47	54	59	43	42	0	1	5.000E-01	.000E+00	.000E+00	.000E+00

***** BOUNDARY ELEMENTS

NUMBER OF ELEMENTS =

20

***** ELEMENT LOAD MULTIPLIERS

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1**** ELEMENT CONNECTIVITY DATA

ELEMENT NUMBER	NODE N	-----DIRECTION I J	NODES----- K L	CODES KD KR	SPECIFIED TRANSLATION	SPECIFIED ROTATION	SPRING RATE
1	1	2 3	2 3	7 1 0	.0000E+00	.0000E+00	1.000E+10
2	2	3 4	3 4	8 1 0	.0000E+00	.0000E+00	1.000E+10
3	3	4 5	4 5	9 1 0	.0000E+00	.0000E+00	1.000E+10
4	4	5 6	5 6	10 1 0	.0000E+00	.0000E+00	1.000E+10
5	5	6 7	6 7	11 1 0	.0000E+00	.0000E+00	1.000E+10
6	6	7 8	7 8	7 1 0	.0000E+00	.0000E+00	1.000E+10
7	7	8 9	8 9	8 1 0	.0000E+00	.0000E+00	1.000E+10
8	8	9 10	9 10	9 1 0	.0000E+00	.0000E+00	1.000E+10
9	9	10 11	10 11	10 1 0	.0000E+00	.0000E+00	1.000E+10
10	10	11 12	11 12	11 1 0	.0000E+00	.0000E+00	1.000E+10
11	11	12 13	12 13	7 1 0	.0000E+00	.0000E+00	1.000E+10
12	12	13 14	13 14	8 1 0	.0000E+00	.0000E+00	1.000E+10
13	13	14 15	14 15	9 1 0	.0000E+00	.0000E+00	1.000E+10
14	14	15 16	15 16	10 1 0	.0000E+00	.0000E+00	1.000E+10
15	15	16 17	16 17	11 1 0	.0000E+00	.0000E+00	1.000E+10
16	16	17 18	17 18	7 1 0	.0000E+00	.0000E+00	1.000E+10
17	17	18 19	18 19	8 1 0	.0000E+00	.0000E+00	1.000E+10
18	18	19 20	19 20	9 1 0	.0000E+00	.0000E+00	1.000E+10
19	19	20 21	20 21	10 1 0	.0000E+00	.0000E+00	1.000E+10
20	20	21 22	21 22	11 1 0	.0000E+00	.0000E+00	1.000E+10

1**** BANDWIDTH MINIMIZATION

MINBND (BANDWIDTH CONTROL PARAMETER) =

1

1**** EQUATION NUMBERS AFTER MINIMIZATION

OLD NODE	NEW NODE	DX	DY	DZ	RX	RY	RZ
1	47	273	0	0	274	275	276
2	54	313	314	315	316	317	318
3	59	343	344	345	346	347	348
4	62	361	362	363	364	365	366
5	60	367	0	0	368	369	370
6	46	267	268	269	270	271	272
7	53	307	308	309	310	311	312
8	58	337	338	339	340	341	342
9	60	349	350	351	352	353	354
10	61	355	356	357	358	359	360
11	48	261	262	263	264	265	266
12	52	301	302	303	304	305	306
13	55	319	320	321	322	323	324
14	56	325	326	327	328	329	330
15	57	331	332	333	334	335	336
16	44	255	256	257	258	259	260
17	48	277	278	279	280	281	282
18	49	283	284	285	286	287	288
19	50	289	290	291	292	293	294
20	51	295	296	297	298	299	300
21	40	211	212	213	214	215	216
22	41	217	218	219	220	221	222
23	42	223	224	225	226	227	228
24	43	229	230	231	232	233	234
25	45	235	236	237	238	239	240
26	47	241	242	243	244	245	246
27	49	247	248	249	250	251	252
28	50	253	254	255	256	257	258
29	52	259	260	261	262	263	264
30	54	265	266	267	268	269	270
31	56	271	272	273	274	275	276

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29	37	213	214	215	216	217	218
30	38	219	220	221	222	223	224
31	39	147	148	149	150	151	152
32	27	153	154	155	156	157	158
33	28	159	160	161	162	163	164
34	30	171	172	173	174	175	176
35	31	177	178	179	180	181	182
36	18	99	100	101	102	103	104
37	19	105	106	107	108	109	110
38	22	123	124	125	126	127	128
39	24	135	136	137	138	139	140
40	25	141	142	143	144	145	146
41	12	63	64	65	66	67	68
42	13	69	70	71	72	73	74
43	15	81	82	83	84	85	86
44	16	87	88	89	90	91	92
45	17	93	94	95	96	97	98
46	6	27	28	29	30	31	32
47	7	33	34	35	36	37	38
48	9	45	46	47	48	49	50
49	10	51	52	53	54	55	56
50	11	57	58	59	60	61	62
51	1	1	0	0	2	3	4
52	2	5	6	7	8	9	10
53	3	11	12	13	14	15	16
54	4	17	18	19	20	21	22
55	5	23	0	0	24	25	26
56	36	207	208	209	210	211	212
57	38	201	202	203	204	205	206
58	29	165	166	167	168	169	170
59	23	129	130	131	132	133	134
60	21	117	118	119	120	121	122
61	20	111	112	113	114	115	116
62	14	75	76	77	78	79	80
63	8	39	40	41	42	43	44
BANDWIDTH BEFORE RESEQUENCING				=		200	
BANDWIDTH AFTER RESEQUENCING				=		60	

1**** NODAL LOADS (STATIC) OR MASSES (DYNAMIC)

NODE NUMBER	LOAD CASE	X-AXIS FORCE	Y-AXIS FORCE	Z-AXIS FORCE	X-AXIS MOMENT	Y-AXIS MOMENT	Z-AXIS MOMENT
56	1	.000E+00	.000E+00	1.631E+03	.000E+00	.000E+00	.000E+00
57	1	.000E+00	.000E+00	4.891E+02	.000E+00	.000E+00	.000E+00
58	1	.000E+00	.000E+00	-5.931E+02	.000E+00	.000E+00	.000E+00
59	1	.000E+00	.000E+00	4.899E+02	.000E+00	.000E+00	.000E+00
60	1	.000E+00	.000E+00	2.450E+02	.000E+00	.000E+00	.000E+00
56	2	.000E+00	.000E+00	1.166E+03	.000E+00	.000E+00	.000E+00
57	2	.000E+00	.000E+00	7.005E+02	.000E+00	.000E+00	.000E+00
58	2	.000E+00	.000E+00	-1.148E+02	.000E+00	.000E+00	.000E+00
59	2	.000E+00	.000E+00	7.005E+02	.000E+00	.000E+00	.000E+00
60	2	.000E+00	.000E+00	3.502E+02	.000E+00	.000E+00	.000E+00

1**** ELEMENT LOAD MULTIPLIERS

LOAD CASE	CASE A	CASE B	CASE C	CASE D
1	.000E+00	.000E+00	.000E+00	.000E+00
2	.000E+00	.000E+00	.000E+00	.000E+00

1**** STIFFNESS MATRIX PARAMETERS

MINIMUM NON-ZERO DIAGONAL ELEMENT = 0-143

MAXIMUM DIAGONAL ELEMENT	=	1.0000E+00
MAXIMUM/MINIMUM	=	3.4380E+04
AVERAGE DIAGONAL ELEMENT	=	5.1471E+08
DENSITY OF THE MATRIX	=	9.9054E+00

!!!! WARNING: MODEL MAY LACK SUFFICIENT BOUNDARY
CONSTRAINT OR HAVE STIFFNESS COMP-
ATIBILITY PROBLEM. SEE NODE AND/OR
ELEMENTS ASSOCIATED WITH DOF:

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1*+++ STATIC ANALYSIS

LOAD CASE =

1

DISPLACEMENTS/ROTATIONS (DEGREES) OF UNRESTRAINED NODES

NODE NUMBER	X- TRANSLATION	Y- TRANSLATION	Z- TRANSLATION	X- ROTATION	Y- ROTATION	Z- ROTATION
1	7.3513E-05	.0000E+00	.0000E+00	-8.3870E-02	-7.5470E-03	.0000E+00
2	7.4615E-05	3.0517E-06	-3.9594E-08	-7.9535E-02	4.4196E-03	.0000E+00
3	7.2321E-05	1.3882E-05	-5.5844E-08	-9.2985E-02	2.5954E-03	.0000E+00
4	6.8430E-05	1.5818E-05	-4.8831E-08	-9.7218E-02	-1.2544E-03	.0000E+00
5	6.3944E-05	.0000E+00	.0000E+00	-1.1112E-01	1.3236E-02	.0000E+00
6	6.6940E-05	1.4989E-06	-1.9985E-08	1.1664E-01	-1.3256E-02	.0000E+00
7	6.7131E-05	8.0463E-06	1.1384E-07	1.3462E-01	6.1206E-03	.0000E+00
8	6.7755E-05	1.4682E-05	6.0398E-08	1.5059E-01	3.4324E-03	.0000E+00
9	6.8697E-05	1.9300E-05	1.5410E-07	1.6607E-01	-3.8462E-03	.0000E+00
10	6.7541E-05	2.0523E-05	1.9399E-08	1.7983E-01	2.4630E-02	.0000E+00
11	5.7904E-05	-5.9457E-07	7.0921E-03	4.1799E-01	-6.6306E-02	.0000E+00
12	5.8336E-05	7.9346E-06	8.0346E-03	4.4697E-01	-4.6935E-02	.0000E+00
13	5.8039E-05	1.5985E-05	8.8589E-03	4.9001E-01	-4.7492E-02	.0000E+00
14	5.7671E-05	2.3855E-05	9.7226E-03	5.3440E-01	-4.6040E-02	.0000E+00
15	5.6549E-05	3.3111E-05	1.0441E-02	5.7752E-01	-2.6397E-02	.0000E+00
16	5.7536E-05	-5.4195E-06	2.3276E-02	4.5537E-01	-8.8704E-02	.0000E+00
17	5.8301E-05	7.0626E-06	2.5030E-02	4.7064E-01	-1.1902E-01	.0000E+00
18	3.8791E-05	2.1810E-05	2.7244E-02	5.0498E-01	-1.3942E-01	.0000E+00
19	3.4713E-05	3.2992E-05	2.9694E-02	5.4319E-01	-1.4667E-01	.0000E+00
20	3.0019E-05	4.6564E-05	3.2286E-02	5.7229E-01	-1.7079E-01	.0000E+00
21	1.0457E-05	-8.2026E-06	3.2969E-02	7.5868E-02	-7.5139E-02	.0000E+00
22	1.0965E-05	2.7003E-06	3.4752E-02	5.3917E-02	-1.3805E-01	.0000E+00
23	1.2523E-05	1.7921E-05	3.7571E-02	1.7758E-02	-2.0027E-01	.0000E+00
24	1.4051E-05	6.4847E-05	4.0732E-02	-1.0176E-02	-1.5021E-01	-8.2065E-02
25	1.3614E-05	3.3323E-05	4.3196E-02	-3.3827E-02	-1.6297E-01	.0000E+00
26	-1.8224E-06	-4.9731E-06	3.1739E-02	-1.8248E-01	-4.0510E-02	.0000E+00
27	-2.0584E-06	-3.4359E-06	3.2702E-02	-1.9144E-01	-5.1819E-02	.0000E+00
28	1.4579E-06	-1.5520E-06	3.3771E-02	-2.4750E-01	-5.0274E-02	.0000E+00
29	1.8184E-05	-2.0326E-06	3.4828E-02	-3.0059E-01	-6.5296E-02	-9.3221E-04
30	3.3180E-05	-1.8027E-06	3.7006E-02	-3.4364E-01	-1.5011E-01	.0000E+00
31	3.3332E-06	5.7971E-07	2.5953E-02	-2.6483E-01	-2.3059E-02	.0000E+00
32	2.1467E-06	-7.5196E-06	2.6499E-02	-2.7226E-01	-3.2807E-02	1.3332E-03
33	6.6379E-06	-2.9818E-05	2.7148E-02	-3.7287E-01	-3.7691E-02	4.8098E-04
34	1.9920E-05	-6.2273E-05	2.7855E-02	-2.6711E-01	-4.4393E-02	3.5423E-03
35	2.4578E-05	-3.7201E-05	2.8735E-02	-2.9649E-01	-5.0492E-02	.0000E+00
36	3.8638E-05	4.7608E-07	1.6061E-02	-2.9284E-01	-3.3672E-02	.0000E+00
37	3.9946E-05	-2.0388E-05	1.6649E-02	-2.9089E-01	-3.1449E-02	.0000E+00
38	4.1833E-05	-5.8857E-05	1.7149E-02	-3.9259E-01	-2.3716E-02	-2.5914E-04
39	3.5792E-05	-4.5567E-05	1.7321E-02	-3.0511E-01	-2.0352E-02	.0000E+00
40	3.1781E-05	-4.9984E-05	1.7325E-02	-3.2709E-01	-1.6928E-02	.0000E+00
41	6.3897E-05	-2.5351E-06	5.4718E-03	-3.9867E-01	-5.5778E-02	.0000E+00
42	6.5538E-05	-1.1960E-05	6.3570E-03	-3.7620E-01	-5.4111E-02	.0000E+00
43	5.7746E-05	-2.0507E-05	7.9192E-03	-3.1631E-01	-1.0351E-02	-5.9027E-04
44	5.1378E-05	-2.7308E-05	6.6244E-03	-3.2745E-01	7.7503E-02	.0000E+00
45	4.9093E-05	-3.5834E-05	6.0928E-03	-3.2589E-01	4.1147E-02	.0000E+00
46	7.1643E-05	-1.6520E-06	-3.9433E-03	-1.0744E-01	-1.0736E-02	.0000E+00
47	7.0730E-05	-3.0238E-06	1.3138E-07	-1.3527E-01	4.7635E-02	.0000E+00

49	6.7034E-05	-1.3217E-05	1.3771E-07	-1.3141E-01	-3.0252E-03	.0000E+00
50	6.4717E-05	-2.1531E-05	-4.3229E-09	-1.1788E-01	1.2376E-02	.0000E+00
51	7.0901E-05	.0000E+00	.0000E+00	7.7458E-02	-6.7262E-03	.0000E+00
52	7.0284E-05	5.0088E-07	-3.7655E-08	7.3018E-02	4.1326E-03	.0000E+00
53	7.3275E-05	-4.6900E-07	-5.2957E-08	8.0659E-02	5.5229E-04	.0000E+00
54	7.4915E-05	-7.0656E-06	-3.9822E-08	7.6068E-02	-3.5310E-03	.0000E+00
55	7.0702E-05	.0000E+00	.0000E+00	8.2798E-02	7.8392E-03	.0000E+00
56	-3.8815E-03	8.7603E-03	4.1320E-02	.0000E+00	-9.7533E-02	-4.9196E-02
57	-2.6934E-03	8.8224E-03	3.4630E-02	.0000E+00	-6.9517E-02	-4.9043E-02
58	-1.3760E-03	9.0114E-03	2.7836E-02	-2.6341E-01	-3.6446E-02	-5.3863E-02
59	-1.3540E-03	9.8431E-03	2.7172E-02	-2.9225E-01	-3.6261E-02	-2.9844E-02
60	-1.3402E-03	1.0179E-02	2.6524E-02	-2.9661E-01	.0000E+00	-1.3563E-02
61	-7.2988E-04	9.9061E-03	1.7116E-02	.0000E+00	-2.0348E-02	-1.1807E-02
62	-3.8531E-04	9.8051E-03	7.1377E-03	.0000E+00	-1.3912E-02	-9.1500E-03
63	-2.3707E-04	9.7494E-03	2.2417E-03	.0000E+00	-1.0683E-02	-8.3267E-03

1**** STATIC ANALYSIS

LOAD CASE =

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DISPLACEMENTS/ROTATIONS(DEGREES) OF UNRESTRAINED NODES

NODE NUMBER	X- TRANSLATION	Y- TRANSLATION	Z- TRANSLATION	X- ROTATION	Y- ROTATION	Z- ROTATION
1	2.3018E-04	.0000E+00	.0000E+00	-1.0089E-01	-6.7391E-03	.0000E+00
2	2.2547E-04	2.7920E-05	-4.7799E-08	-9.6000E-02	5.5083E-03	.0000E+00
3	2.1793E-04	4.0426E-05	-5.9134E-08	-1.1308E-01	3.3616E-03	.0000E+00
4	2.0738E-04	4.1564E-05	-5.9749E-08	-1.1893E-01	-1.4011E-03	.0000E+00
5	1.9637E-04	.0000E+00	.0000E+00	-1.3623E-01	1.6329E-02	.0000E+00
6	2.0803E-04	1.5211E-05	-2.6173E-08	1.3923E-01	-1.3672E-02	.0000E+00
7	2.0798E-04	2.9342E-05	1.3670E-07	1.6247E-01	7.5409E-03	.0000E+00
8	2.0981E-04	4.2856E-05	7.3145E-08	1.8314E-01	4.4464E-03	.0000E+00
9	2.1229E-04	4.7352E-05	1.8878E-07	2.0320E-01	-4.5716E-03	.0000E+00
10	2.0976E-04	4.8657E-05	2.4792E-08	2.2123E-01	3.0486E-02	.0000E+00
11	1.9321E-04	1.7400E-05	8.4995E-03	5.0304E-01	-9.4833E-02	.0000E+00
12	1.9333E-04	3.2275E-05	9.7175E-03	5.4146E-01	-6.0949E-02	.0000E+00
13	1.9280E-04	4.6696E-05	1.0787E-02	5.9721E-01	-6.1532E-02	.0000E+00
14	1.9207E-04	6.0386E-05	1.1908E-02	6.5493E-01	-6.2351E-02	.0000E+00
15	1.8931E-04	7.6542E-05	1.2852E-02	7.1126E-01	-3.5461E-02	.0000E+00
16	1.5802E-04	1.4213E-05	2.8081E-02	5.5496E-01	-1.1951E-01	.0000E+00
17	1.5857E-04	3.5893E-05	3.0404E-02	5.7641E-01	-1.5550E-01	.0000E+00
18	1.5846E-04	6.0972E-05	3.3289E-02	6.2131E-01	-1.8137E-01	.0000E+00
19	1.3133E-04	8.1016E-05	3.6486E-02	6.7255E-01	-1.9124E-01	.0000E+00
20	1.4324E-04	1.0531E-04	3.9873E-02	7.1373E-01	-2.2671E-01	.0000E+00
21	1.1106E-04	1.4620E-05	4.0128E-02	1.0567E-01	-1.3728E-01	.0000E+00
22	1.1112E-04	3.3408E-05	4.2562E-02	8.3943E-02	-1.3233E-01	.0000E+00
23	1.1230E-04	6.1304E-05	4.6262E-02	3.9976E-02	-2.6063E-01	.0000E+00
24	1.1200E-04	1.3807E-04	5.0494E-02	8.9955E-03	-2.1191E-01	-7.2166E-02
25	1.1557E-04	9.3081E-05	5.3975E-02	-1.3227E-02	-2.2546E-01	.0000E+00
26	9.1919E-05	2.3131E-05	3.9074E-02	-1.8070E-01	-8.9259E-02	.0000E+00
27	9.0254E-05	2.6186E-05	4.0572E-02	-2.1244E-01	-8.5674E-02	.0000E+00
28	9.1905E-05	3.3088E-05	4.2228E-02	-2.7554E-01	-8.1547E-02	.0000E+00
29	1.1233E-04	5.3588E-05	4.3973E-02	-3.3680E-01	-1.3932E-01	-2.2367E-02
30	1.3949E-04	2.9536E-05	4.7222E-02	-3.8968E-01	-2.1765E-01	.0000E+00
31	1.0318E-04	3.8140E-05	3.2215E-02	-3.2233E-01	-3.3513E-02	.0000E+00
32	1.0547E-04	1.8837E-05	3.3516E-02	-3.3505E-01	-6.6619E-02	4.0612E-03
33	1.1601E-04	-1.1749E-05	3.4637E-02	-3.3313E-01	-8.0091E-02	1.3717E-02
34	1.3868E-04	-6.9597E-05	3.6040E-02	-3.1477E-01	-8.3715E-02	5.2538E-03
35	1.5048E-04	-3.9749E-05	3.7640E-02	-3.9675E-01	-9.3323E-02	.0000E+00
36	1.6745E-04	1.0760E-05	2.0089E-02	-3.6403E-01	-8.8973E-02	.0000E+00
37	1.7336E-04	-2.7120E-05	2.1394E-02	-3.5663E-01	-9.7110E-02	.0000E+00
38	1.6416E-04	-9.8716E-05	2.2026E-02	-3.7577E-01	-4.6771E-02	2.2621E-02
39	1.7089E-04	-7.0659E-05	2.2577E-02	-3.7577E-01	-5.1579E-02	.0000E+00
40	1.7632E-04	-7.8294E-05	2.3077E-02	-3.7577E-01	-4.7314E-02	.0000E+00

41	2.1748E-04	-1.0448E-05	3.3439E-05	-3.1028E-01	-7.6583E-02	1.0000E+00
42	2.2009E-04	-3.4248E-05	8.0659E-05	-3.9115E-01	-7.5593E-02	1.0000E+00
43	2.1133E-04	-6.5220E-05	9.0472E-05	-4.0688E-01	-3.0541E-02	-1.2651E-03
44	2.0573E-04	-6.1199E-05	8.7442E-05	-4.2250E-01	4.1352E-02	1.0000E+00
45	2.0415E-04	-6.8966E-05	8.1160E-05	-4.3362E-01	4.5497E-02	1.0000E+00
46	2.3389E-04	-1.4322E-05	-1.4557E-08	-1.3273E-01	-1.3317E-02	1.0000E+00
47	2.3307E-04	-2.6475E-05	1.5179E-07	-1.5974E-01	5.3831E-03	1.0000E+00
48	2.3148E-04	-2.9151E-05	4.3825E-08	-1.7679E-01	-2.0178E-03	-1.8344E-03
49	2.2904E-04	-4.4426E-05	1.8332E-07	-1.7202E-01	-3.3457E-03	1.0000E+00
50	2.2578E-04	-4.7614E-05	-2.8071E-09	-1.5684E-01	1.6728E-02	1.0000E+00
51	2.4888E-04	1.0000E+00	1.0000E+00	9.8282E-02	-8.3237E-03	1.0000E+00
52	2.4390E-04	-2.1942E-03	-4.7906E-08	9.3309E-02	5.5363E-03	1.0000E+00
53	2.3878E-04	-3.1157E-05	-6.8396E-08	1.0347E-01	1.1000E-03	1.0000E+00
54	2.3129E-04	-3.5238E-05	-5.2179E-08	9.9493E-02	-4.3175E-03	1.0000E+00
55	2.1997E-04	1.0000E+00	1.0000E+00	1.0903E-01	1.0589E-02	1.0000E+00
56	-6.0584E-03	9.8827E-03	5.1109E-02	1.0000E+00	-1.6621E-01	-5.2373E-02
57	-4.7268E-03	1.0013E-02	4.3721E-02	1.0000E+00	-1.3524E-01	-5.9463E-02
58	-2.6699E-03	1.0351E-02	3.6082E-02	-2.9649E-01	-8.6540E-02	-1.1019E-01
59	-2.8367E-03	1.2302E-02	3.4677E-02	-3.7505E-01	-6.9184E-02	-7.3964E-02
60	-2.6155E-03	1.3152E-02	3.3345E-02	-3.8567E-01	1.0000E+00	-3.5587E-02
61	-1.2582E-03	1.2534E-02	2.1962E-02	1.0000E+00	-3.5504E-02	-2.2903E-02
62	-6.1831E-04	1.2492E-02	9.1994E-03	1.0000E+00	-2.5467E-02	-1.7095E-02
63	-3.4156E-04	1.2432E-02	2.9223E-03	1.0000E+00	-1.9546E-02	-1.5583E-02

1**** THIN PLATE/SHELL ELEMENT STRESSES

ELEMENT CASE --		MEMBRANE STRESS COMPONENTS --			--- BENDING STRESS COMPONENTS --		
NO.	(MODE)	SM11	SM22	SM12	SB11	SB22	SB12

1	1	-6.501E+00	1.249E+01	2.172E+01	-3.690E+03	-1.392E+04	9.108E+02
1	2	-2.444E+01	1.535E+02	8.675E+01	-3.236E+03	-1.674E+04	1.140E+03
2	1	2.906E+00	-2.043E+01	1.778E+01	-4.379E+03	-2.077E+04	-3.369E+03
2	2	1.580E+01	5.425E+01	5.256E+01	-5.291E+03	-2.509E+04	-4.537E+03
3	1	5.492E+00	-3.951E+01	4.627E+00	-1.048E+03	-1.763E+03	-1.748E+03
3	2	1.169E+01	6.631E+00	8.842E+00	-1.321E+03	-2.457E+03	-2.388E+03
4	1	3.154E+00	-5.100E+01	-2.121E+01	1.331E+03	1.925E+04	-2.590E+02
4	2	4.822E+00	-1.362E+01	-3.757E+01	1.549E+03	2.281E+04	-4.504E+02
5	1	-4.891E+00	-2.947E+01	-2.464E+01	6.736E+02	1.537E+04	2.552E+03
5	2	-2.132E+01	6.349E+00	-2.712E+01	7.695E+02	1.866E+04	2.857E+03
6	1	-2.083E+00	1.348E+01	-5.456E+00	7.951E+02	5.913E+03	9.109E+02
6	2	2.812E+00	-2.207E+01	6.335E+01	5.910E+02	8.537E+03	6.600E+02
7	1	-1.315E+01	-9.799E+01	4.511E+01	7.556E+01	1.127E+03	-3.333E+01
7	2	-1.307E+01	-4.646E+02	1.053E+02	-1.172E+02	1.703E+03	-2.525E+02
8	1	1.731E+01	4.461E+01	-1.944E+01	4.133E+02	6.027E+02	-1.410E+02
8	2	3.767E+01	-1.977E+02	-6.833E+01	4.823E+02	8.749E+02	-1.484E+02
9	1	-2.168E+00	8.540E+01	-1.730E+01	-2.951E+03	-1.266E+04	3.636E+02
9	2	1.122E+01	4.588E+01	-9.576E+01	-3.719E+03	-1.595E+04	5.046E+02
10	1	-5.002E+00	7.737E+01	-4.233E+00	-3.614E+03	-1.235E+04	-9.825E+02
10	2	-3.222E+01	1.726E+02	-9.412E+01	-3.278E+03	-1.630E+04	-1.267E+03
11	1	-2.406E+01	4.853E+01	2.472E+01	-5.009E+03	-1.538E+04	3.047E+01
11	2	-7.882E+01	1.339E+01	4.999E+01	-6.058E+03	-1.936E+04	5.540E+01
12	1	3.697E+00	1.472E+01	1.308E+01	-6.968E+03	-2.271E+04	-2.310E+03
12	2	4.376E+01	7.388E+01	7.346E+01	-9.412E+03	-2.763E+04	-3.887E+03

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13	2	3.280E+01	1.395E+02	2.781E+01	-1.841E+03	-1.953E+03	-3.803E+03
14	1	1.287E+01	-5.811E+01	1.755E+01	2.747E+03	3.223E+04	-3.471E+03
14	2	1.978E+01	-9.883E+00	3.532E+01	2.983E+03	2.637E+04	-3.134E+03
15	1	-7.262E-01	-2.477E+02	5.774E+00	2.704E+03	1.703E+04	5.689E+03
15	2	-5.986E+01	-3.805E+02	3.877E+01	3.051E+03	2.027E+04	5.709E+03
16	1	2.447E+01	-3.055E+02	-7.901E+01	1.441E+03	3.782E+03	4.716E+02
16	2	2.467E+01	-5.067E+02	8.516E+00	2.018E+03	6.308E+03	-1.241E+02
17	1	1.470E+00	-3.034E+02	-1.354E+02	4.398E+02	1.047E+03	5.598E+02
17	2	2.950E+01	-9.259E+02	-1.789E+02	5.226E+02	1.915E+03	1.356E+03
18	1	1.673E+01	3.442E+02	-1.463E+02	2.995E+03	1.872E+03	-1.178E+03
18	2	8.611E+01	1.953E+02	-3.496E+02	3.847E+03	2.475E+03	-1.228E+03
19	1	-3.228E+01	3.506E+02	7.426E+01	-1.801E+03	-1.200E+04	3.152E+03
19	2	-2.523E+01	4.152E+02	-6.445E+01	-1.380E+03	-1.531E+04	4.400E+03
20	1	6.552E+00	-4.417E+01	6.902E+01	-4.811E+03	-1.458E+04	-1.961E+02
20	2	-9.894E+01	-5.248E+00	1.091E+00	-6.231E+03	-1.874E+04	-3.077E+02
21	1	-3.653E+01	2.078E+01	1.499E+01	-5.866E+03	-1.779E+04	4.378E+02
21	2	-1.007E+02	4.922E+01	1.985E+01	-7.167E+03	-2.171E+04	5.666E+02
22	1	3.101E+01	7.560E+01	-1.305E+01	-7.789E+03	-2.473E+04	-3.725E+03
22	2	3.101E+01	1.674E+02	-3.709E+01	-9.542E+03	-3.025E+04	-3.524E+03
23	1	-3.508E+01	9.795E+01	-1.144E+01	-5.902E+02	-7.407E+02	-3.461E+03
23	2	-4.245E+01	2.403E+02	-2.114E+01	-8.835E+02	-1.255E+03	-4.513E+03
24	1	2.620E+01	2.106E+02	1.932E+02	1.037E+04	2.780E+04	-2.914E+03
24	2	1.069E+01	4.193E+02	3.029E+02	1.178E+04	5.304E+04	-3.891E+03
25	1	1.567E+01	-8.300E+02	2.333E+02	6.579E+03	1.953E+04	9.419E+03
25	2	-4.652E+01	-1.103E+03	4.660E+02	6.435E+03	2.288E+04	1.133E+04
26	1	1.961E+02	-7.967E+02	-1.580E+02	-2.253E+03	-9.416E+02	-1.296E+02
26	2	1.478E+02	-1.590E+03	-1.523E+01	-2.831E+03	2.778E+02	-1.136E+02
27	1	8.593E+01	-6.364E+01	3.532E+01	2.903E+02	1.464E+03	1.165E+03
27	2	9.169E+01	-5.831E+02	1.266E+02	5.500E+02	3.134E+03	2.188E+03
28	1	-6.200E+01	3.918E+02	1.241E+02	3.009E+03	1.398E+03	2.369E+03
28	2	-6.127E+01	2.727E+02	3.089E+02	3.428E+03	2.364E+03	3.684E+03
29	1	1.148E+00	4.105E+02	-5.493E+01	-1.470E+03	-1.219E+04	-2.078E+03
29	2	4.033E+01	5.229E+02	9.684E+01	-1.854E+03	-1.576E+04	-1.275E+03
30	1	-1.372E+01	-2.490E+01	-1.094E+02	-4.741E+03	-1.488E+04	1.621E+01
30	2	-1.353E+02	2.886E+01	-7.246E+01	-6.106E+03	-1.929E+04	-2.206E+01
31	1	-1.657E+01	2.175E+02	-5.143E+01	-3.540E+03	-1.839E+04	-2.177E+02
31	2	-4.215E+01	5.137E+02	-1.568E+02	-4.335E+03	-3.280E+04	-1.396E+02
32	1	2.150E+01	1.818E+02	-1.700E+01	-3.375E+03	-3.534E+04	-1.821E+03
32	2	4.687E+01	4.096E+02	-5.336E+01	-5.571E+03	-3.177E+04	-2.171E+03
33	1	-3.363E+01	1.522E+02	-1.069E+01	-1.590E+03	-1.523E+03	-4.633E+03
33	2	-5.378E+01	3.434E+02	-1.331E+01	-2.919E+03	-5.642E+03	-6.036E+03
34	1	4.341E+01	1.477E+02	-1.805E+02	7.146E+03	3.345E+04	5.275E+02
34	2	3.559E+01	3.746E+02	-3.012E+02	8.555E+03	7.559E+04	5.175E+02

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35	1	1.372E+01	1.701E+02	1.000E+02	2.210E+03	1.700E+04	1.700E+04
35	2	1.785E+01	-1.426E+03	-2.949E+02	2.752E+03	2.370E+04	-2.750E+03
36	1	8.369E+00	-9.230E+02	1.156E+02	-4.541E+03	-3.919E+03	4.791E+03
36	2	7.029E+00	-1.860E+03	1.700E+02	-5.158E+03	-3.086E+03	5.950E+03
37	1	1.936E+01	3.426E+01	1.795E+02	3.971E+02	1.747E+03	3.739E+02
37	2	3.539E+01	-2.837E+02	3.055E+02	9.129E+02	4.023E+03	6.456E+02
38	1	-2.304E+01	2.299E+02	2.101E+01	5.079E+02	5.757E+02	2.599E+03
38	2	-3.326E+01	1.347E+02	5.896E+01	5.015E+02	8.111E+02	4.040E+03
39	1	1.589E+01	2.768E+02	2.372E+01	-3.189E+03	-1.364E+04	-2.874E+03
39	2	4.448E+01	3.821E+02	1.059E+02	-4.236E+03	-1.795E+04	-3.316E+03
40	1	-1.584E+01	2.629E+02	4.460E+01	-2.858E+03	-1.376E+04	8.690E+02
40	2	-5.134E+01	5.337E+02	1.655E+02	-3.780E+03	-1.813E+04	1.073E+03
41	1	8.807E+02	3.087E+03	2.030E+03	-3.692E+02	-2.276E+03	-1.420E+03
41	2	1.355E+03	3.032E+03	3.233E+03	-5.170E+02	-1.760E+03	-1.573E+03
42	1	8.482E+02	-1.320E+03	-1.277E+03	-3.102E+02	-8.435E+02	-2.394E+03
42	2	1.766E+03	-1.018E+03	-1.056E+03	-1.999E+03	-6.396E+02	-3.729E+03
43	1	-1.202E+02	1.973E+01	2.731E+01	1.360E+03	-9.740E+01	-2.906E+03
43	2	8.540E+00	5.400E+02	-1.947E+02	2.017E+03	-1.408E+02	-6.456E+03
44	1	-2.493E+01	3.671E+02	3.156E+02	7.459E+02	-1.164E+03	-1.331E+03
44	2	-1.674E+01	4.586E+02	4.941E+02	1.849E+03	-2.371E+03	-3.432E+03
45	1	2.557E+02	3.208E+01	-5.997E+02	5.491E+02	3.612E+00	-7.122E+02
45	2	1.120E+03	1.264E+02	-1.052E+03	1.484E+03	-2.551E+02	-1.439E+03
46	1	-2.847E+02	3.629E+02	-8.963E+02	8.077E+01	2.402E+01	-4.064E+02
46	2	1.216E+02	6.771E+02	-1.547E+03	8.431E+01	-1.740E+02	-7.332E+02
47	1	-5.028E+02	-2.849E+02	1.303E+03	8.424E+01	3.700E+02	-3.213E+02
47	2	-4.114E+02	-2.516E+01	1.265E+03	1.378E+02	6.284E+02	-5.848E+02

I**** BOUNDARY ELEMENT FORCES/MOMENTS

ELEMENT NO.	CASE (MODE)	FORCE	MOMENT
1	1	.0000E+00	.0000E+00
1	2	.0000E+00	.0000E+00
2	1	-3.9594E+02	.0000E+00
2	2	-4.7794E+02	.0000E+00
3	1	-5.6844E+02	.0000E+00
3	2	-6.9134E+02	.0000E+00
4	1	-4.8631E+02	.0000E+00
4	2	-5.9749E+02	.0000E+00
5	1	.0000E+00	.0000E+00
5	2	.0000E+00	.0000E+00
6	1	-1.9925E+02	.0000E+00
6	2	-2.6173E+02	.0000E+00
7	1	1.1704E+03	.0000E+00
7	2	1.3670E+03	.0000E+00

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	8	1	7.3145E+02	.0000E+00
	9	1	+1.5410E+03	.0000E+00
	9	2	+1.8875E+03	.0000E+00
	10	1	1.9399E+02	.0000E+00
	10	2	2.4792E+02	.0000E+00
46	11	1	-9.9439E+01	.0000E+00
	11	2	-1.4557E+02	.0000E+00
47	12	1	1.2138E+03	.0000E+00
	12	2	1.5178E+03	.0000E+00
48	13	1	5.6027E+01	.0000E+00
	13	2	4.3625E+02	.0000E+00
49	14	1	+1.3771E+03	.0000E+00
	14	2	+1.3332E+03	.0000E+00
50	15	1	-4.3229E+01	.0000E+00
	15	2	-2.8071E+01	.0000E+00
51	16	1	.0000E+00	.0000E+00
	16	2	.0000E+00	.0000E+00
52	17	1	-3.7835E+02	.0000E+00
	17	2	-4.7908E+02	.0000E+00
53	18	1	-3.2657E+02	.0000E+00
	18	2	-6.8396E+02	.0000E+00
54	19	1	-3.9822E+02	.0000E+00
	19	2	-5.2179E+02	.0000E+00
55	20	1	.0000E+00	.0000E+00
	20	2	.0000E+00	.0000E+00

***** TEMPORARY FILE STORAGE (MEGABYTES)

UNIT NO.	7 :	.023
UNIT NO.	8 :	.033
UNIT NO.	9 :	.029
UNIT NO.	10 :	.023
UNIT NO.	11 :	.164
UNIT NO.	12 :	.009
UNIT NO.	13 :	.305
UNIT NO.	14 :	.050
UNIT NO.	15 :	.023
UNIT NO.	17 :	.000
TOTAL	:	.659

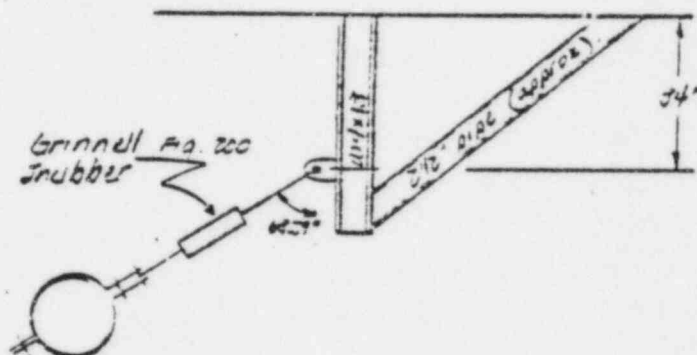
***** END OF FILE

CC: ALBOS

SI-01-6007-H00J

LOADS:

$$|W + P_d| + (JSE^2 + AM^2)^{1/2} \\ = 583 \text{ lbs.}$$



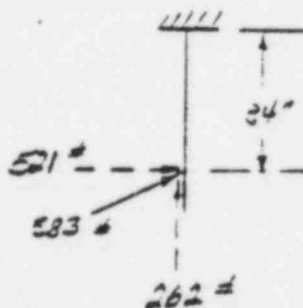
CONSIDER TRUBBER, CLAMP, CLEVISES:

ASSUME all Grinnell parts: (due to lack of info.)

<u>Part</u>	<u>Allowable Load (lbs)</u>
200	10,350
295	2,365
66	3,770

∴ All parts are acceptable

CONSIDER VERTICAL WIDE FLANGE:



$$A = 3.73 \\ J = 5.46 \\ r = 1.00$$

TENSION: $F_t = 21,600 \text{ psi}$, LEVEL $D = 40,608 \text{ psi}$

$$\sigma_t = \frac{262}{3.73} = 68 \text{ psi} < 21,600 \text{ psi}$$

Compression:

$$C_c = 126$$

$$\frac{KL}{r} = \frac{2.0(34)}{1.00} = 68 < 126$$

$$F_c = \frac{\left[1 - \frac{(68)^2}{2(126)^2}\right] 36,000}{\frac{5}{3} + \frac{7(68)}{8(126)} - \frac{(68)^2}{8(126)^2}} = 16,631 \text{ psi}$$

$$\text{LEVEL } \sigma = 31,266 \text{ psi}$$

$$\sigma_a = \frac{262}{3.83} = 68 \text{ psi} < 16,631 \text{ psi}$$

$$\text{Shear: } F_v = 14,400 \text{ psi}, \quad \text{LEVEL } \sigma = 37,072 \text{ psi}$$

$$\sigma_v = \frac{521}{3.83} = 136 \text{ psi}$$

$$\text{Bending: } F_b = .66(36,000) = 23,760 \text{ psi}$$

$$\text{LEVEL } \sigma = 44,669 \text{ psi}$$

$$\sigma_b = \frac{521(34)}{5.45} = 3244 \text{ psi} < 23,760 \text{ psi}$$

CONSIDER WELDS:

clevis to wide Flange - $\frac{1}{4}$ " fillet all around

$$\text{Tension: } \sigma_t = \frac{521}{.707(\frac{1}{4})(4)} = 737 \text{ psi}$$

$$\text{Shear: } \sigma_v = \frac{262}{.707(\frac{1}{4})(4)} = 371 \text{ psi}$$

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wide flange to existing large beam - 1/4" fillet all around

TENSION: $\sigma_t = \frac{262}{.707 (1/4) (21)} = 71 \text{ psi}$

shear: $\sigma_v = \frac{521}{.707 (1/4) (21)} = 140 \text{ psi}$

bending: $\sigma_b = \frac{521 (34)}{.707 (1/4) [2(4)(4) + \frac{(4)^2}{3}]} = 2684 \text{ psi}$

TOTAL: $\underline{\hspace{1cm}} 2895 \text{ psi}$

$\therefore$ This support is acceptable

SI-01-6007-H002

LOADS:

$$|W + P_d| = 685 \text{ lb.}$$

$$|W + P_d + T| = 731 \text{ lb.}$$

$$|W + P_d + T| + (\overline{J}E^2 + AM^2)^{1/2} = 1266 \text{ lbs.}$$

CHECK WIDE FLANGES: $A = 3.83$, $J = 5.46$

$$\text{shear: } F_v = .4(35) = 14.4 \text{ ksi}$$

$$\text{LEVEL D} = 27.07 \text{ ksi}$$

$$\sigma_v = \frac{731}{3.83} = 191 \text{ psi} < 14,400 \text{ psi}$$

$$\sigma_v = \frac{1266}{3.83} = 331 \text{ psi} < 27,070 \text{ psi} \quad \text{LEVEL D}$$

$$\text{bending: } F_b = .66(35) = 23.75 \text{ ksi}, \quad \text{LEVEL D} = 44.67 \text{ ksi}$$

$$\sigma_b = \frac{1266(12^{3/4})}{5.46} = 2956 \text{ psi} < 23,760 \text{ psi}$$

$$\text{TENSION: } F_t = .6(35) = 21.6 \text{ ksi}, \quad \text{LEVEL D} = 40.61 \text{ ksi}$$

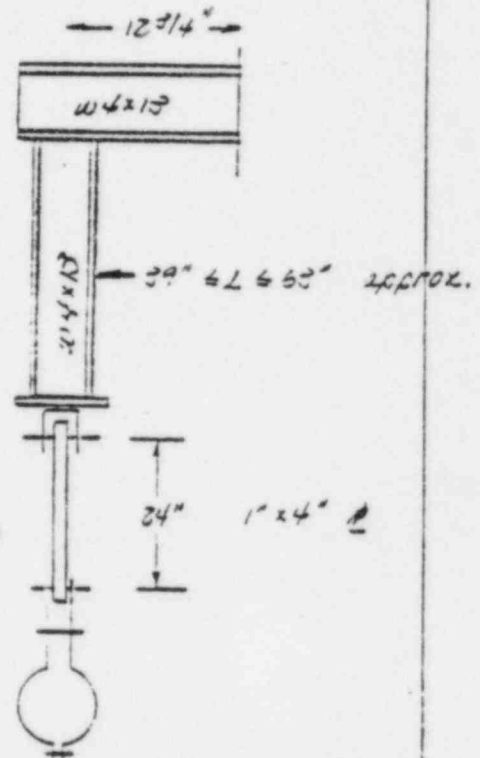
$$\sigma_t = \frac{1266}{3.83} = 330 \text{ psi} < 21,600 \text{ psi}$$

COMPRESSION:

$$C_c = \sqrt{\frac{2\pi^2(70,000)}{25,000}} = 125$$

$$\frac{L}{r} = \frac{10(1.57)}{1.00} = 15.7 < 125$$

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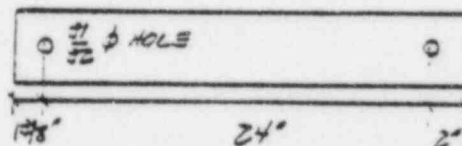
$$F_2 = \frac{\left[1 - \frac{63.0^2}{2(126)^2}\right] (36,000)}{\frac{1}{3} + \frac{2(63.0)}{8(126)} - \frac{(63.0)^2}{8(126)^2}} = 17133 \text{ psi}$$

$$\text{LEVEL } D = 32210 \text{ psi}$$

$$\sigma_a = \frac{1266}{3.83} = 330 \text{ psi} < 17133 \text{ psi}$$

CHECK 1" x 4" PL:

1" HOLE



1" HOLE

$$\text{shear: } F_v = 14.4 \text{ ksi}$$

$$\text{LEVEL } D = 27.07 \text{ ksi}$$

$$\sigma_v = \frac{1266}{1(1.518)} = 920 \text{ psi} < 14,400 \text{ psi}$$

$$\text{TENSION (tear out): } F_t = 21.6 \text{ ksi, LEVEL } D = 40.61 \text{ ksi}$$

$$\sigma_t = \frac{1266}{(1)(4 - 2(1.518))} = 418 \text{ psi} < 21,600 \text{ psi}$$

compression:

$$C_c = 126$$

$$r = \sqrt{(2(1.518/12))/(1)(4)} = 0.042$$

$$\frac{L}{r} = \frac{2(24)}{0.042} = 1143 > 126 \text{ (USE 200)}$$

$$F_c = \frac{12\pi^2 E}{25(L/r)^2} = \frac{12\pi^2 (29 \times 10^6)}{25(200)^2} = 3733 \text{ psi}$$

$$\text{LEVEL } D = 7019 \text{ psi}$$

$$\sigma_c = \frac{1266}{2(1)} = 633 \text{ psi} < 3733 \text{ psi}$$

CHECK CLAMP & CLEVIS:

Assume Grinnell parts:

<u>Part No.</u>	<u>maximum load (lbs.)</u>
295	5390
66	7090

All parts are acceptable.

CHECK WELDS:

CLEVIS to 6x6" R → 1/4" Fillet L = 3 3/4"

$$\sigma_f = \frac{1266}{.707(1/4)(3 3/4)} = 1910 \text{ PSI}$$

8 wide Flanges → 1/4" Fillet

$$\sigma_f = \frac{1266}{.707(1/4)(2)(2 3/4)} = 1802 \text{ PSI}$$

$$\sigma_f = \frac{1266}{.707(1/4)(2)(2 3/4)} = 1802 \text{ PSI}$$

$$\sigma_b = \frac{1266(12 3/4)}{(.707)(1/4)(2)(2 3/4)(11)} = 8856 \text{ PSI}$$

∴ this support is acceptable.

SUPPORT CALCULATIONSSI-01-6007-H00L

LOADS: $|W + P_3 + T| = 442 \text{ lbs}$

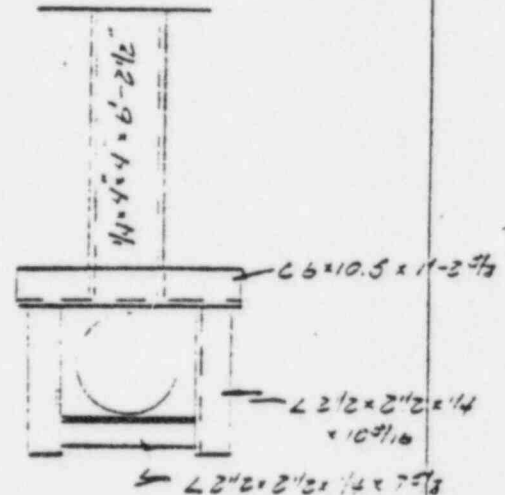
$$|W + P_3 + T| + (35E^2 + Am^2)^{1/2} = 747 \text{ lbs.}$$

CONSIDER VERTICAL TUBE:

TENSION: $F_p = .5(36,000) = 21,500 \text{ psi}$

LEVEL 0 = 40,608 psi

$$\sigma_p = \frac{747}{3.59} = 208 \text{ psi} < 21,500 \text{ psi}$$



Compression: $C_2 = \sqrt{\frac{2\pi^2(36 \times 10^6)}{36,000}} = 126$

$$\frac{L}{r} = \frac{2(38'12")}{1.51} = 51.0 < 126$$

$$F_2 = \frac{\left[1 - \frac{(51.0)^2}{2(126)^2}\right] 36,000}{\frac{5}{3} + \frac{2(51)}{3(126)} - \frac{(51)^2}{3(126)^2}} = 18,259 \text{ psi}$$

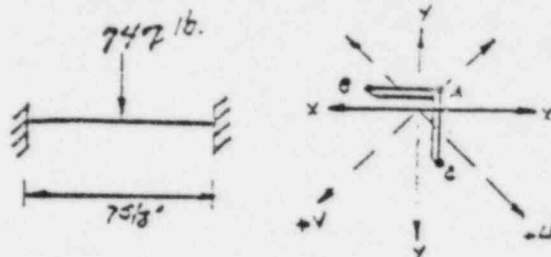
LEVEL 0 = 34,326 psi

$$\sigma_2 = \frac{747}{3.59} = 208 \text{ psi} < 18,259 \text{ psi}$$

CONSIDER HORIZONTAL X:

Consider the member as a fixed-fixed beam w/ a concentrated load at center -

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Shear: $F_v = .4(36,000) = 14,400 \text{ psi}$, Level A = 27,072 psi

$$\tau_v = \frac{747}{1.19} = 628 \text{ psi} < 14,400 \text{ psi}$$

Bending: $F_b = .6(36,000) = 21,600 \text{ psi}$, Level A = 40,608 psi

$$d = 1.19 \text{ in.}$$

$$x_o = y_o = 0.717 \text{ in.}$$

$$I_x = I_y = 0.705 \text{ in}^4$$

$$r_u = 0.941 \text{ in.}$$

$$I_u = r_u^2 = 1.19(.941)^2 = 1.054 \text{ in}^4$$

$$I_u + I_v = I_x + I_y \quad I_v = 2(.705) - 1.054 = 0.352 \text{ in}^4$$

$$v_A = -1.014$$

$$u_A = 0$$

$$v_B = 0.754$$

$$u_B = -1.768$$

$$v_C = +0.754$$

$$u_C = 1.768$$

$$M_u = -.707(747)(75 \frac{1}{8})/8 = -503 \text{ in-lb}$$

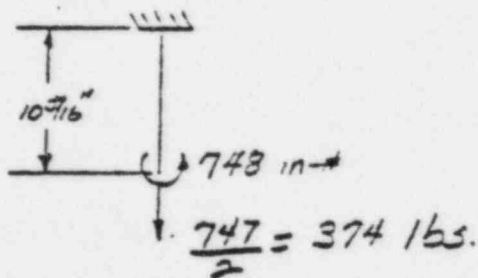
$$M_v = .707(747)(75 \frac{1}{8})/8 = 503 \text{ in-lb}$$

$$\sigma_A = \frac{M_u v_A}{I_u} + \frac{M_v u_A}{I_v} = \frac{-503(-1.014)}{1.054} = 484 \text{ psi} < 21,600 \text{ psi}$$

$$\sigma_B = \frac{-503(.754)}{1.054} + \frac{503(-1.768)}{.352} = -2466 \text{ psi} < 21,600 \text{ psi}$$

$$\sigma_C = \frac{-503(.754)}{1.054} - \frac{503(1.768)}{.352} = 2288 \text{ psi} < 21,600 \text{ psi}$$

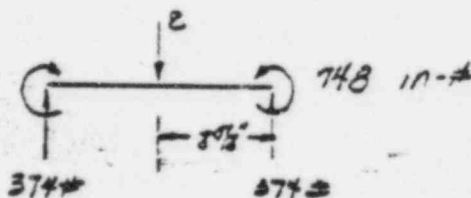
CONSIDER VERTICAL & J:



Tension: $\sigma_t = \frac{374}{1.19} = 314 \text{ psi} < 21,600 \text{ psi}$

Bending: calculations performed for horizontal & $2' \frac{1}{2} \times 2' \frac{1}{2} \times \frac{1}{4}$ would apply here also.

CONSIDER CHANNEL:



$A = 3.09$
 $S = .564$

Shear: $\sigma_v = \frac{374(2)}{3.09} = 242 \text{ psi} < 14,400 \text{ psi}$

Bending: Consider half of beam as cantilever —

$\sigma_b = \frac{374(35 \frac{1}{2}) + 748}{.564} = 7046 \text{ psi} < 40,603 \text{ psi}$
LEVEL 0

$\sigma_b = \frac{442(1/2)(35 \frac{1}{2}) + 442(7 \frac{1}{2})/3}{.564} = 4,127 \text{ psi} < 21,600 \text{ psi}$

CONSIDER WELDS:

Tube to existing beam: $\frac{1}{4}$ " fillet

$$\text{Tension: } \sigma_t = \frac{747}{.707(1/4)(4)(3)} = 352 \text{ psi}$$

Horizontal Δ to vertical Δ : $\frac{1}{4}$ " fillet

$$\text{shear: } \sigma_v = \frac{747}{.707(1/4)(2)(2\frac{1}{2})} = 845 \text{ psi}$$

$$\text{bending: } \sigma_b = \frac{748}{.707(1/4) \left[\frac{4(2\frac{1}{2})(2\frac{1}{2})^2 - (2\frac{1}{2})^3}{6} \right]} = 813 \text{ psi}$$

(see "Design of welded structures", Blodgett)

vertical Δ to channel: $\frac{1}{4}$ " fillet

bending: same as above calculation

$$\text{Tension: } \sigma_t = \frac{374}{.707(1/4)(4)(2\frac{1}{2})} = 212 \text{ psi}$$

channel to vertical tubing: $\frac{3}{16}$ " fillet

$$\text{Tension: } \sigma_t = \frac{747}{.707(3/16)(2)(4)} = 704 \text{ psi}$$

$$\text{bending: } \sigma_b = \frac{748(2)}{.707(3/16)(4)(4)} = 705 \text{ psi}$$

$\therefore$ This support is acceptable

05/11/25.

NUPIPE-TIL. NUCLEAR SERVICES CORPORATION PIPING ANALYSIS PROGRAM - VERSION 1
 SERIES 1 ST-150 PIPE STRESS CALCULATION W IMPELL SAMS

SUPPORT REACTIONS FOR LOAD CASE NO. 1

HEIGHT PLUS EXTERNAL FORCES

NODE	TYPE	REACTION (LBS OR IN-LBS)	DIRECTION
5	FORCE	-9.	X COORD
5	FORCE	100.	Y COORD
5	FORCE	-1.	Z COORD
5	MOMENT	15.	X COORD
5	MOMENT	72.	Y COORD
5	MOMENT	-5733.	Z COORD
20	FORCE	408.	Y COORD
35	FORCE	686.	Y COORD
85	FORCE	466.	Y COORD
85	FORCE	1.	INCLINED
95	FORCE	210.	Y COORD
95	FORCE	-41.	INCLINED
105	FORCE	196.	Y COORD
105	FORCE	36.	Z COORD
115	FORCE	198.	Y COORD
115	FORCE	-4.	Z COORD
120	FORCE	209.	Y COORD
120	FORCE	-25.	Z COORD
135	FORCE	10.	Y COORD
135	FORCE	-3.	INCLINED
250	FORCE	39.	X COORD
250	FORCE	-231.	Y COORD
250	FORCE	19.	Z COORD
250	MOMENT	2916.	X COORD
250	MOMENT	-566.	Y COORD
250	MOMENT	1978.	Z COORD

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P5/11/25.

NUPIPE-TIL. NUCLEAR SERVICES CORPORATION PIPING ANALYSIS PROGRAM - VERSION 1
 SPACS 1 ST-150 PIPE STRESS CALCULATION W IMPELL SAPS

SUPPORT REACTIONS FOR LOAD CASE NO. 2

THERMAL EXPANSION PLUS ARCHING MOVEMENTS

NODE	TYPE	REACTION (LBS OR IN-LBS)	DIRECTION
5	FORCE	-178.	X CCCC
5	FORCE	-11.	Y CCCC
5	FORCE	-46.	Z CCCC
5	MOMENT	579.	X CCCC
5	MOMENT	-5555.	Y CCCC
5	MOMENT	151.	Z CCCC
20	FORCE	34.	Y CCCC
35	FORCE	45.	Y CCCC
85	FORCE	-145.	Y CCCC
85	FORCE	54.	INCLINED
95	FORCE	-82.	Y CCCC
95	FORCE	49.	INCLINED
105	FORCE	6.	Y CCCC
105	FORCE	-67.	Z CCCC
115	FORCE	-69.	Y CCCC
115	FORCE	-102.	Z CCCC
120	FORCE	427.	Y CCCC
120	FORCE	176.	Z CCCC
135	FORCE	-1036.	Y CCCC
135	FORCE	-100.	INCLINED
250	FORCE	156.	X CCCC
250	FORCE	667.	Y CCCC
250	FORCE	-124.	Z CCCC
250	MOMENT	-11105.	X CCCC
250	MOMENT	-2393.	Y CCCC
250	MOMENT	-50093.	Z CCCC

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SUPPORT REACTIONS FOR LOAD COMBINATION CASE NO. 24

SEISMIC PLUS IMPSANS

NODE	TYPE	REACTIONS (LBS OR IN-LBS)	DIRECTION
5	FORCE	879.	X COORD
5	FORCE	127.	Y COORD
5	FORCE	327.	Z COORD
5	MOMENT	952.	X COORD
5	MOMENT	21449.	Y COORD
5	MOMENT	7365.	Z COORD
20	FORCE	305.	Y COORD
35	FORCE	535.	Y COORD
40	FORCE	563.	Y COORD
65	FORCE	403.	Y INCLINED
65	FORCE	306.	Y COORD
95	FORCE	100.	Y INCLINED
95	FORCE	373.	Y COORD
105	FORCE	83.	Y INCLINED
105	FORCE	602.	Y COORD
115	FORCE	149.	Z COORD
115	FORCE	166.	Z COORD
120	FORCE	166.	Z COORD
120	FORCE	137.	Z COORD
135	FORCE	446.	Y COORD
135	FORCE	393.	Y COORD
155	FORCE	862.	Y INCLINED
160	FORCE	010.	Y INCLINED
165	FORCE	125.	Y INCLINED
250	FORCE	1653.	X COORD
250	FORCE	705.	Y COORD
250	FORCE	1733.	Z COORD
250	MOMENT	81452.	X COORD
250	MOMENT	77406.	Y COORD
250	MOMENT	72245.	Z COORD

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