

PHILADELPHIA ELECTRIC COMPANY

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SHIELDS L. DALTROFF
VICE PRESIDENT
ELECTRIC PRODUCTION

(215) 841-5001
September 28, 1984

Docket Nos. 50-277
50-278

Mr. John F. Stolz
Operating Reactors Branch #4
Division of Licensing
U.S. Nuclear Regulatory Commission
Washington, D.C. 20555

SUBJECT: Peach Bottom Atomic Power Station, Units 2
and 3, Request for Relief of Operating Restrictions
on the Purge and Vent Valves

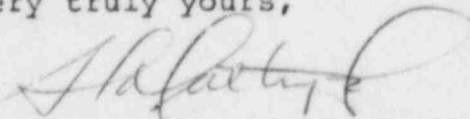
REFERENCE: Correspondence dated April 24, 1984,
S. L. Daltroff (PECo) to J. F. Stolz (NRC)

Dear Mr. Stolz:

This letter formally transmits design data for the six and eighteen-inch containment purge and vent valves as requested by the NRC staff in a September 5, 1984, telephone conversation. The data consists of valve actuator torque output calculations as previously reviewed by the NRC's consultants. It is our understanding that the data was found to be acceptable to the consultant. This data relates to our request in the referenced correspondence to increase the maximum allowable open position on the containment purge and vent valves based on planned improvements to valve components.

Should you have any questions regarding this matter, please do not hesitate to contact us.

Very truly yours,



8410040421 840928
PDR ADOCK 05000277
PDR

cc: A. R. Blough, Site Inspector

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Peach Bottom Atomic Power Station
Units 2 & 3

Calculations for Actual Actuator Torque Output for
6" & 18" Fisher Continental Division Butterfly Valves, 9200 Type

Serial Nos. 136937 thru 136954

Provided by: Bruce D. Hatton
Clifford B. Ives & Co., Inc.
Representative of Fisher Controls International, Inc.

Equation for the actual Actuator Torque Output:

$$T_N = M (Pa - \Delta_T B_S - 1.05 Kt)$$

where T_N = Actual (Net) Torque Output

M = Moment Arm @ linkage

P = Max. Air Pressure

a = Diaphragm Area @ desired \angle

Δ_T = Initial Diaphragm Area

B_S = Initial Spring Compression Pressure

1.05 = Constant

K = Spring Constant

t = Spring Travel Distance

Calculation of Actuator Torque Output for the
6" Fisher Butterfly Valves, 9200 Type, S/N 136953 & 136954

656 Size 40 Net Torque Per Degree of Rotation

$$\begin{aligned} T_N &= M (Pa - \Delta_T B_S - 1.05 Kt) && \text{Spring 1L2173} \\ &= M (30a - 99.7(6) - 1.05 335t) && \text{Size 40} \\ &= M (30a - 598.2 - 351t) && \text{30 Psi Supply} \\ & && \text{6.26 Bench} \\ & && M, a, t \text{ from Table 1} \end{aligned}$$

$$\begin{aligned} 10^\circ T_N &= 2.02 (30 \times 86 - 598.2 - 351 \times .33) \\ &= 2.02 (1981.8 - 115.83) \\ &= 3769.25 \end{aligned}$$

$$\begin{aligned} 20^\circ T_N &= 2.25 (30 \times 78.5 - 598.2 - 351 \times .70) \\ &= 2.25 (1756.8 - 245.7) \\ &= 3399.97 \end{aligned}$$

$$\begin{aligned} 30^\circ T_N &= 2.40 (30 \times 74.5 - 598.2 - 351 \times 1.11) \\ &= 2.40 (1636.8 - 389.61) \\ &= 2993.25 \end{aligned}$$

$$\begin{aligned} 40^\circ T_N &= 2.46 (30 \times 72.6 - 598.2 - 351 \times 1.53) \\ &= 2.46 (1579.8 - 537.03) \\ &= 2565.21 \end{aligned}$$

$$\begin{aligned} 50^\circ T_N &= 2.45 (30 \times 72.3 - 598.2 - 351 \times 1.96) \\ &= 2.45 (1570.8 - 687.96) \\ &= 2162.95 \end{aligned}$$

$$\begin{aligned} 60^\circ T_N &= 2.37 (30 \times 72.5 - 598.2 - 351 \times 2.39) \\ &= 2.37 (1576.8 - 838.89) \\ &= 1748.84 \end{aligned}$$

$$\begin{aligned} 70^\circ T_N &= 2.23 (30 \times 72.6 - 598.2 - 351 \times 2.79) \\ &= 2.23 (1579.8 - 979.29) \\ &= 1339.13 \end{aligned}$$

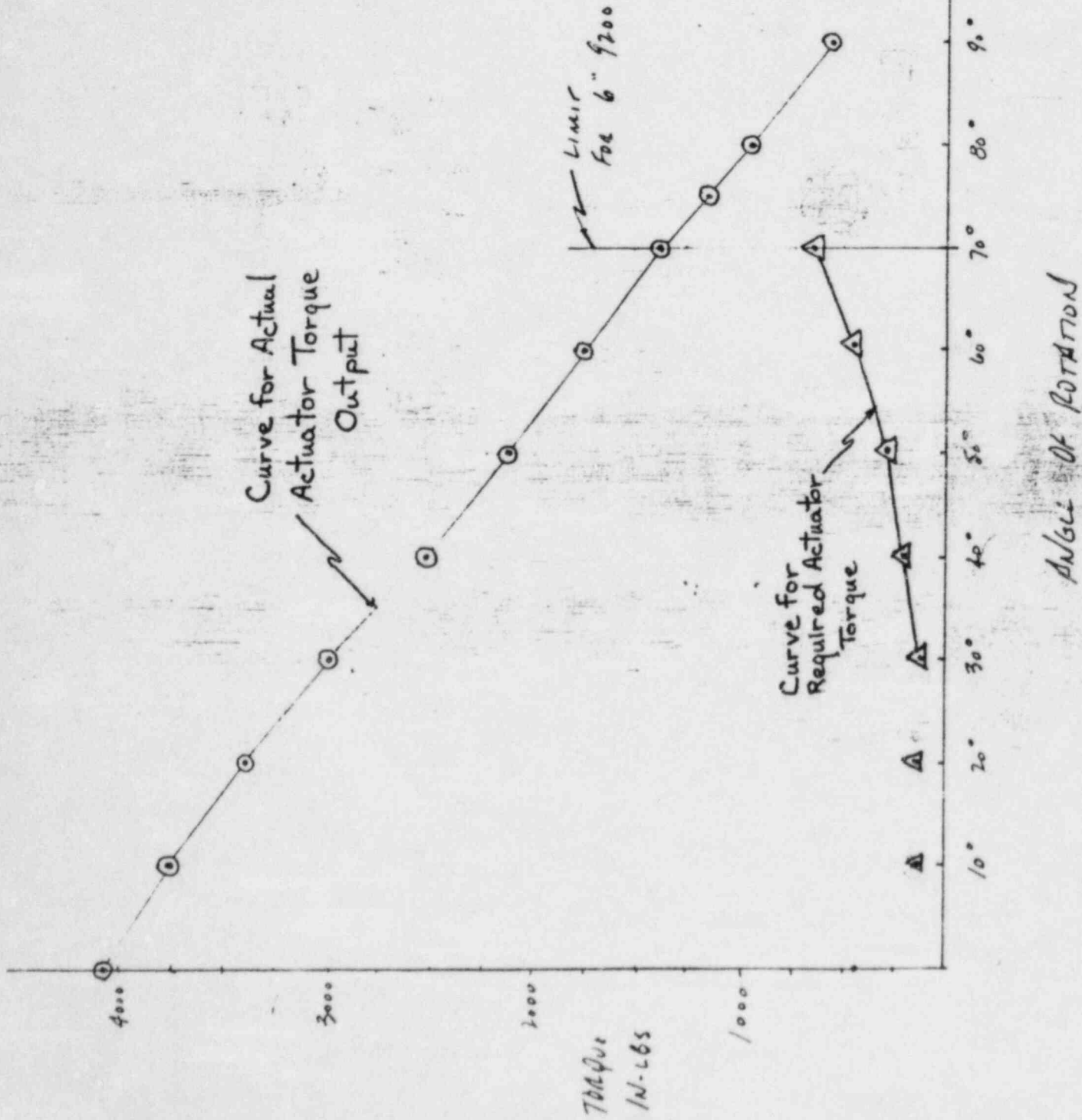
$$\begin{aligned} 80^\circ T_N &= 2.02 (30 \times 72.2 - 598.2 - 351 \times 3.16) \\ &= 2.02 (1567.8 - 1109.16) \\ &= 926.45 \end{aligned}$$

$$\begin{aligned} 90^\circ T_N &= 1.77 (30 \times 70.4 - 598.2 - 351 \times 3.49) \\ &= 1.77 (1513.8 - 1224.99) \\ &= 511.19 \end{aligned}$$

$$\begin{aligned} 0^\circ T_N &= 1.72 (30 \times 99.7 - 598.2) \\ &= 4115.61 \end{aligned}$$

$$\begin{aligned} 75^\circ T_N &= 2.13 (30 \times 72.5 - 598.2 - 351 \times 2.98) \\ &= 2.13 (1576.8 - 1045.98) \\ &= 1130.64 \end{aligned}$$

DIAPHRAGM TORQUE
 AVAILABLE TO OVERCOME
 DYNAMIC TORQUE
 6" Valve, 9200 Type
 656 SIZE 40 Actuator



Calculation of Actuator Torque Output for the
18" Fisher Butterfly Valves, 9200 Type, S/N 136937 thru 136952

656 Size 60 Net Torque Output

Spring 1K1628
30 psi Supply
6.26 Bench

$$T_N = M (P_a - \Delta_T B_S - 1.05 Kt)$$
$$= M (30a - 214.9(6) - 1.05 (610)t)$$

$$0^\circ T_N = 2.06 (30 \times 214.9 - 214.9(6) - 1.05 \times 610 \times 0)$$
$$= 2.06 (5157.6)$$
$$= 10624.65$$

$$10^\circ T_N = 2.41 (30 \times 181.5 - 1289.4 - 640.5 \times .39)$$
$$= 2.41 (4155.6 - 249.79)$$
$$= 9413.0$$

$$20^\circ T_N = 2.67 (30 \times 171.2 - 1289.4 - 640.5 \times .84)$$
$$= 2.67 (3846.6 - 538.02)$$
$$= 8833.90$$

$$30^\circ T_N = 2.85 (30 \times 165.5 - 1289.4 - 640.5 \times 1.32)$$
$$= 2.85 (3675.6 - 845.46)$$
$$= 8065.89$$

$$40^\circ T_N = 2.93 (30 \times 164.5 - 1289.4 - 640.5 \times 1.82)$$
$$= 2.93 (3645.6 - 1165.71)$$
$$= 7266.07$$

$$50^\circ T_N = 2.92 (30 \times 164.2 - 1289.4 - 640.5 \times 2.34)$$
$$= 2.92 (3636.6 - 1498.77)$$
$$= 6242.46$$

$$60^\circ T_N = 2.83 (30 \times 164.5 - 1289.4 - 640.5 \times 2.84)$$
$$= 2.83 (3645.6 - 1819.02)$$
$$= 5169.22$$

$$70^\circ T_N = 2.65 (30 \times 164.4 - 1289.4 - 640.5 \times 3.32)$$
$$= 2.65 (3642.6 - 2126.46)$$
$$= 4017.77$$

