# [50 Nuclear 

## Oyster Creek Evaluation of Thermomag Fire Barriers

Report 102<br>REV. 0

Project No:/TDR No.: BA 328348

Date
RASOD: BARADARAN

## Approvals:


$\frac{12 / 4 / 95}{\text { DATE }}$

$\frac{12-12-95}{\text { DATE }}$



#### Abstract

The purpose of this report is to provide the methodology for establishing the fire endurance rating of installed Thermo-Lag fire barrier raceway systems. This report summarizes the results of the evaluations which establish the aforementioned fire endurance ratings and identifies those Thermo-Lag fire barrier raceway syetems which meet the requirements of Appendix $R$, section IIIC, those barriers which do not meet Appendix $R$ and will be modified or upgraded to nieet Appendix $R$, and those barriers which do not meet Appendix $R$ and for which an evaluation will be performed to justify the fire endurance rating in an exemption request.

This report also provides the methodology used to evaluate the hazards in each fire area or fire zone where Thermo-Lag fire berrier raceway systems are installed. These hazard evaluations will be documented in exemption requests and will serve as the basis for supporting such exemptions where the fire endurance rating of the Thermo-Lag fire barrier raceway system does not meet the requirements of Appendix R, Section IIIG.


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### 1.0 PURPOSE

The purpose of this report is to provide the methodology for establishing the fire endurance rating (equivalent rating by test comparison) of installed Thermolag fire barrier raceway systems at oyster Creek. Fire endurance rating is established by identifying the "Actual Fire Rating" or the rating consistent with the fire endurance test acceptance critaria as defined in NRC Generic Letter 8610 Supplement 1, "Fire Endurance Test Acceptance Criteria for Fire Barrier Systems Used to Separate Redundant Safe Shutdown Trains Within the Same Fire Area". Fire endurance rating is almo vetablished by tdentifying the "Cable Qualification Rating" or the rating which, based upon establishing the maximum temperature inside a fire barrier envelofs that is considered acceptable to demonstrate cable functionality. Use of the "Cable qualification Rating" is a deviation from the GL 86-10 acceptance criteris but is acceptable based upon an engineering evaluation as described herein. Results of these evaluations are reported in section 3.

This report also provides the methodology used to evaluate the hazards in each fire area or fire zone. The hazard evaluation will serve as the basi. for Eupporting exemptions from Appendix R, Section IIIG.

### 2.0 METHODOLOGY

### 2.1 Establishing Actual Fire Rating

To assess material performance and provide a basis for evaluation of installed Thermo-Lag fire barriers, an industry fire endurance test program was conducted by the Nuclear Energy Institute (NEI). To address issues with the fire endurance capability of installed barrier configurations, the industry test program:

- Assessed current industry configurations through the use of survey data,
- Conducted tests to establish performance of various baseline and upgraded fire barrier system assemblies, end
- Developed a guideline to assist utilities in evaluating installed barrier configurations.

The guideline developed by NEI is known as the "NEI Application Guide for Evaluation of Thermo-Lag 330 Fire Barrier Systems" (Report no, 0784-00001-TR-02 Revision 1) or the "Application Guide". The Application Guide provides a process and data for evaluation of installed Thermo-Lag fire barrier configurations using information obtained from NEI and utility fire endurance test programs. GPU Nuclear has used this process to

- Establish the extent that installed barrier configurations can be bounded by previous tests,
- Determine the fire endurance capability (or "Actual Fire Rating") that installed barrier configurations, which are bounded by test, can be reasonably expected to provide, and
- Propose upgrades to installed barrier configurations where deemed necessary to achieve an acceptable fire rating.

In order to evaluate the extent that installed barrier conditions at oyster Creek can be bounded by test configurations and data in the Application Guide, GPU Nuclear performed the following:

- A walkdown of the fire areas/zones was conducted to document the installed barrier configurations with digitized computer images.
- The parameters identified by NEI during the industry fire endurance test program which pertain to fire enduranca capability as identified in the Appiication Guide were included by GPU Nuclear in an electronic database.
- Each fire barxier system was separated into individual segments or elements for evaluation purposes. Individual elements are constituted by one or more of the following distinguishing characteristice:

1) change in barrier construction technique;
2) significant change in protected raceway or contents;
3) variation from applicable barrier installation requirements;
4) change in type of barrier material; or
5) change in orientation of protected raceway or change which necessitates a change in barrier construction technique.

- Data collected during the walkdown and collected by a review of the original fire barrier construction details were entered into the database to permit detailed comparisons of relevant parameters from the NEI, Texas Utilities (TU), and TVA programs.
- Each of the test assemblies in the industry test programe was separated into individual segments or elements for evaluation purposes as were the installed fire barrier systems and entered into the data base in order to permit the detailed comparisons of relevant parameters with the installed fire barrier systems.
- The quality of the barrier installation was originally verified by quality Control during the installation process by continuous in-process inspections using checklists for consistency. Final inspections performed by GPUN Quality Control were documented in the work control packages. The use of checklists standardized the attribute verification and allowed no significant deviations from the original design/ingtallation requirements. Repairs are performed to the same requirements as the initial installation. Repairs are performed either by or under the supervision of certified installers and are re-inspected by certified inspectors. Surveillance procedures ensure a refueling interval inspection to reverify the integrity of the installed fire barrier envelopes. Additionally, QC does make inspections of opportunity and has caused repairs to be initiated.

A Quality Control Program as defined in the Oyster Creek FHAR-Section 5 (and implemented by the GPUN Operational Quality Assurance Plan) was applied to the material, processes and instaliation and inspection personnel. GPUN did not contract this work out to a third party licensed by TSI. GPUN contracted ISI to train and certify GPUN personnel for both installation and inspection.

GPUN's initial experience with TSI's material shipments resulted in returning large amounts of each shipment. All inspections were documented on receipt inspection reports. All line items of the contract were inspected. These inspections consisted of either 100 sampling using engineering approved inspection and sampling plans (Military Standard 105D was the prescribed sampling method). Some of the problems experienced were as follows:

THICKNESS
GPUN receipt inspectcrs verified thickness by taking readings on each piece. Any piece not meeting the minimum thickness was rejected and returned to TSI.

VOIDS AND CRACKS (resulting from bending/fabrication)
During receipt inspection, material which had voids or cracking beyond epecified limits was returned. The material was subject to considerable field work (cut and fit). Any voids were filled with TSI trowel grade material during installation.

## PHYSICAL DAMAGE

During receipt inepection, gouges, crushing and chipped edges could be repaired and accepted. Separation of the material from the stress skin was a cause for rejection and return of the material. Edges were verified to be straight and square for proper alignment and fit-up during installation.

To ensure receipt of consistent quality material from TSI, GPUN instituted QC checks by GPUN vendor surveillance at TSI's factory prior to release for shipment. Thorough receiving inspection checks were performed to established inspection plans to insure they maintained engineering requirements.

## DETAILED EXAMINATIONS

GPU Nuclear performed detailed exams to confirm the accuracy of Quality Assurance records for important parameters which are not visible by walkdown. The initial exam consisted of dismantling a 3 hour conduit barrier. Additional exams as committed in GPUN letter c321-95-2277 dated September 22, 1995 were performed on 1 hour and 3 hour conduit barriers, 1 hour box configurations and the HVAC duct berrier. Parameters such as thickness, panel sib orientation, stress skin orientation, buttering of joints, joint gap width, unsupported spans and type of joint were confirmed and documented in Quality Verification Inspection Report PIR $\# 950065$ dated october 25, 1995. The reaults of these inspections confirmed conformance of installed Thermo-Lag with original installation and design requirements and provides a reasonable basis for reliance on Quality Assurance records and installation requiremente for parameters which are not visible by walkdown.

In order to establish the actual fire rating of the installed fire barrier assemblies, the industry test data was evaluated. Acrual fire rating is a term used to designate the fire endurance rating of the barrier consistent with the acceptance criteria contained in NFPA 251 (ASTM E-119), "Standard Fire Tests of Building Construction and Materials". NRC Gereric Letter 86-10 Supplement 1, adapts the acceptance criteria of NFPA 251 to casle raceway fire barrier wraps.

In 86-10, Supp. 1, the staff bases acceptability of a fire endurance qualification test for fire barrier materials applied directly to a raceway or component to be successful if the "average" unexposed side temperature of the fire barrier system, as measured on the exterior surface of the raceway or component did not exceed 250 deg $F$ above its initial temperature and a visual inspection of cables inside the raceway should show no signs of degraded conditions. Aiso, individual temperature readings should not exceed the $250 \mathrm{deg} F$ temperature rise by more than 30 percent, or 325 deg F above the initial temperature.

To establish the barrier rating (ACTUAL RATING) of a test assembly, GPU Nuclear reviewed the temperature data for the test and identified that point in time when the first individual temperature reading on the unexposed gide of the fire barrier for the entire raceway in the test assembly, as measured on the exterior surface of the raceway or component, exceeded 325 deg $F$ above the initial temperature. Note that this method establishes e rating for all elements of a particular raceway size based upon the weakest link in the raceway. While it is possible to establish individual ratings in a test involving straight conduit, radial bends and condulets based upon thermocouple readings restricted to these elements in a test assembly, it is conservative to establish a common rating for all elements of a raceway based upon the single high reading for the entire raceway.

To establish the actual rating for an installed configuration or element, the installed configuration's relevant parameters were compared with those of the industry tested configurations. If an acceptable match was found, it was selected and the installed configuration is considered bounded by an acceptable industry test configuration. The actual rating of the matching industry configuration becomes the actual rating of the installed configuration or element as documented by a detailed evaluation. The results of these detailed comparisons and evaluations will be listed in this eubmittal only. These results will be retained in the electronic databace ( DOC. No. TLDB-OC-814-1) and digitized computer image library mentioned previously and will be available for NRC review or audit.

### 2.2 Establishing Cable qualification Rating

In addition to establishing the "Actual Fire Rating" as previously described, GPU Nuclear has utilized a combination of actual test data and theoretically derived data to document raceway temperatures inside the Thermo-Lag fire barrier envelopes which deviate from the acceptance criteria used for qualifying fire barrier configurations as outlined in Generic Letter $86-10 \mathrm{Supp}$. 1 . This section outlines the method that serves as the basis for establishing the fire endurance rating of the barrier when considering the cable qualification temperature or the maximum temperature inside the fire barrier envelope that is considered acceptable to demonstrate cable functionality. The acceptable fire endurance rating is termed the "Cable Qualification Rating". GL 86-10 Supp. 1 permits an evaluation which demonstrates that cables would perform their intended function during and after a postulated fire exposure when the internal raceway temperatures exceed the GL $86-10$ Supp. 1 acceptance criteria. GPU Nuclear considers the Cable Qualification Rating directly comparable to the requirement of III.G for one hour or three hour fire barriers. In other words, a fire barrier having a cable qualification rating of at least 60 minutes is considered to be a one hour fire barrier as required by III.G. If less than 60 minutes, the cable qualification rating can be used to establish the basis for an exemption from the requirement for a one hour fire barrier.

In order to compare the internal raceway temperatures to cable failure temperatures, GPU Nuclear performed the following:

## 2.2a Identification of Protected Circuits

The Oyster Creek Fire Hazards Analysis Report (FHAR) Document No. 990-1746 Revision 7 was compared with the Thermo-Lag Installation Drawings to identify those circuits required for safe shutdown which are protected by the Thermo-Lag fire barriers.

## 2. 2b Identification of Cable Qualification Temperatures (Note this is not to be confused with environmental qualification temperatures)

Generic Letter 86-10, Supplement 1, Attachment to Enclogure 1, "Acceptable Methods for Demonstrating Functionality of Cables Protected by Raceway Fire Barrier Systems During and After Fire Endurance Test Exposure" provides the means for establishing the maximum temperature inside the fire barrier envelope that is considered acceptable to demonstrate cable functionality, GPU Nuclear used the recommended analysis of section VI in the aforementioned attachment entitled "Cable Thermal Exposure Threshold" to perform this analysis.

After identifying the circuits required for safe shutdown as described in " 2.2 a " above, the cable manufacturer was identified and the thermal exposure threshold (TET) temperature limit or insulation failure threshold temperature limit was obtained from Sandia Test Report SAND90-0696 May, 1991 "An Investigation of the Effects of Thermal Aging on the Fire Damageability of Electrical Cables". Insulation damage leading to a short circuit per the tests conducted in the aforementioned investigation is defined as 15 milliamps ( mA ) leakage current with cable conductor temperatures at 90 deg c which means that the cable was tested at the equivalent of full load current. The insulation failure threshold temperature iimit is the temperature beyond which the insulation is expected to degrade causing a short circuit. While not the same as the short circuit rating as defined in Generic Letter 86-10, Supplement 2, these tests conclusively demonstrate cable functionality at elevated temperatures since under fire conditions the endurance rating of the barrier is based upon the point in time when the threshold temperature is reached. The aforementioned Sandia tests on the other hand, subjected cables to the insulation failure threshold temperature for at least 80 minutes without failure. Fire endurance ratings do not assume sustained operation under these elevated temperature conditions. The rating is based upon the duration of the test and the point in the time of the test when the insulation failure threshold temperature is reached. This method is conservative because, as stated above, the Sandia teste subjected cables to the insulation failure threshold temperature for AT LEAST 80 MINUTES.

## 2.2c Raceway Temperatures Inside Thermo-Lag Fire Barrier Envelopes

In order to follow the guidance of Generic Letter $86-10$, Supplement 1 for establishing maximum allowable raceway temperatures inside the fire barrier envelope, additional evaluation of industry test data and consideration of the operating cable temperatures within the fire barrier system at the onset of the fire exposure were performed. This is necessary in order to compare test results with the TET temperature limit and establish a "cable qualification rating" for the fire barrier envelope.

Industry test data which was used to establish the "actual" fire rating described previousiy was further evaluated to document internal raceway temperatures beyond NRC acceptance criteria as defined in GL 86-10. To be consistent, the evaluation was limited to the temperature readings on the unexposed side of the fire barrier as measured on the exterior surface of the raceway. The evaluation documented the temperatures and the duration of the fire test at the aforementioned locations in increments of approximately 100 deg $F$ up to the point where the test was terminated. eg. for $3 / 4 \mathrm{in}$. conduit in NEI test $2-1$, the actual rating is based upon a conduit surface temperature of 387 deg $F$ at 27 minutes. Since the test was continued beyond this point, the following data was utilized to compare conduit surface temperatures with the insulation failure threshold temperature mentioned in 2.2b.
3/4" Conduit Outside Surface Temp (Maximum)

| TIME (MIN) | TEMP (DEG |
| :---: | :---: |
| 28 | 396 |
| 36 | 499 |
| 41 | 609 |
| 44 | 720 |

By reviewing the temperature data in this case it is apparent that the barrier failed to provide any protection after about 44 minutes. In the case of the $3 / 4^{\prime \prime}$ conduit, Thermo-Lag material was consumed directly exposing the conduit to test oven temperatures. According to NEI test report 2-1, no joint failures occurred.

If the test was terminated before a duration of 60 minutes, it was necessary to develop a multi-dimensional heat transfer model to extrapolate temperatures inside the raceway out to 60 minutes for the $2^{\prime \prime}, 4^{\prime \prime}$ and $6^{\prime \prime}$ conduit assemblies since no temperature data was avillable. The model was verified against actual industry test data to establish confidence in ite validity. Comperison with test data on NEI "upgraded" conduit raceway configurations (NEI test 1-6) was performed. This is considered legitimate since the upgrades consisted of joint reinforcement which was accomplished by adding stress skin and trowel grade material at joints; however, the thickness of the barrier remained the same at $1 / 2^{\prime \prime}$ except in the areas where the foints were reinforced. Burnthrough did not occur on the $1 / 2^{\prime \prime}$ Thermo-lag after 60 minutes. It is also assumed that the baseline joints of NEI test $2-1$ would not have failed had the tegt run the full 60 minutes. As stated above, joint failure of the $3 / 4^{n}$ conduit configurations did not occur. It is reasonable to assume that joint failure would not have occurred on the larger size raceways in NEI test $2-1$ and that burnthrough would not have occured on $1 / 2^{\prime \prime}$ thick fire barriers on these raceways because another test (NEI Test $1-6$ ) on the same nominal thickness for larger size raceways that went the full 60 minutes did not burnthrough. It is therefore considered reasonable to extrapolate internal raceway temperatures for NEI Test 2-1 out to 60 minutes for the purpose of comparison with cable qualification temperatures. The results of this model are documented in calculation c-9000-814-5310-002. This calculation is not included here but is available for NRC review or audit.

It is necessary to account for the initial temperature of the cable within the fire barrier prior to the onset of the fire as compared to the test configurations. As such, the cable qualification rating is established based upon the time it takes for the internal raceway temperature to get within no less than 70 deg $F$ of the insulation failure threshold temperature. This 70 deg $F$ value provides the margin to account for the differences between actual room ambient temperatures plus cable temperature rise due to the insulating effects of the barrier and those ambient temperatures measured during industry testing. The industry tests were performed with initial ambient temperatures no less than 50 deg $F$ while the maximum design room ambient temperatures are $104 \mathrm{deg} F$ at Oyster Creek. The additional temperature rise inside the fire barrier envelopes due to operating cables is 7 deg $F$ maximum based upon test results documented in letter G/C/TMI-1CS/16503 dated September 15, 1988, J. Brendlen to J.W. Langenbach, "TSI Derating Check". While this test was conducted for installed cable tray barriers at TMI-1, it is reasonable to apply these results to oyster Creek since the barrier thickness at Oyster Creek is the same as at TMI-1. The main factor impacting heat rise inside a raceway would be the thickness of the barrier and the number of circuits energized inside the raceway. The TMI-1 test was conducted in trays containing continuously energized power circuite which can be considered comparable to raceways at Oyster Creek. Both TMI-1 and Oyeter Creek have barriers with a nominal thickness ranging from minimual of $1 / 2$ to $5 / 8$ inch for one hour barriers and 1 to $1-1 / 8$ inch for three hour barriers. Therefore using TMI-1 test results to factor into initial raceway temperatures at Oyster Creek is reasonable. The additional effect of the internal temperature of the raceway due to continuously energized power circuits is only about 10 of the factor being used. Adding the difference between the max room ambient
temperature and the minimum ambient test temperature ( $104-50=54$ deg $F$ ) to the temperature $r$ ise of 7 deg $F$ yivids a total factor of 61 deg $F$. The factor being used is 70 deg $F$. This eetalishes that the cable qualification rating is consistent with maximum cable operaiing temperatures. This meets the guidance in GL 86-10 for an engineering analysis to demonstrate the functionality of cables inside fire barrier envelopes.

### 2.3 Evaluating Fixe Eazards

To evaluate the cable qualification ratings of the Thermo-Lag fire barriers which do not meet the requirements of Appendix R Section III.G (ie. less than 1 hour or 3 hour), two methods are mployed. These methode consider the insitu and transient hazards in a fire area/zone. These are described as follows:

## 2.3a Method 1

The first method is typical of the "tracitional approach". A comparison of the cable qualification rating with the cvorall fire loading is performed using $80,000 \mathrm{BTU} / \mathrm{Ft} 2$ as equivalent to a one houi fire to develop a fire load to rating factor hereir referred to as the rating fictor. A combustible loading of 80,000 BTU/Ft2 has been considered as equivalent to the heat release in an ASTM E-119 Test Oven for a one hour fire duration test. A source reference to support this assumption is Table 6-6a of the NFPA Fire Protection Handbook, Seventeenth Edition. If the Thermo-Lag fire barrier is located in afire area/zone which is provided with an area wide fire suppression system, the aforementioned factor is multiplied by an additional factor of 3 to account for the additional marcin provided by the suppression system. This factor is an assumption based upor, the fact that Appendix R Section III.G.2.c reduces the requirement for $a$; hour barrier in III.G.2. a to 1 hour with the presence of a suppression and decection system; hence the additional factor of 3 .

The rating factor is used for assessing the fire hazard with the actual rating of a raceway fire barrier that does not meet the reguirements of Appendix R. No strict acceptance criteria is established as the acceptability of a rating factor must consider all fire hazard and fire protection features where the raceway fire barrier is located. Acceptability is therefore based upon engineering judgement.

The following reference point will be taken into consideration in judging the acceptability of a rating factor. The basis for this is drawn from the oyster Creek FHAR, paragraph 1.3 "Delineation of Fire Areas/Zones". To summarize, this section of the FHAR established that reasonable assurance that a fire will not propagate from one fire zone to another is provided in the plant fire hazards analysis by passive and active fire protection features. Through this assurance, it can be justified that fire zones which may not consist of continuous wall to wall, floor to ceiling boundaries of fire rated construction can be analyzed by themselves. Fire zones are subdivisions of fire areas which take into consideration the physical boundaries which exist between one fire zone and another in the same fire area. Fire zone boundaries can consist of non-rated physical boundaries with penetrations sealed with materials installed in accordance with configurations similar to a one hour minumum fire barrier. The criferia for acceptability of such a configuration is that the combustible loading on either side of the boundary is less than 40,000 BTU/Ft2, or $1 / 2$ the equivalent of a one hour fire duration as discussed above or a rating factor of 2 .

The following example is provided to illustrate how the cable qualification rating factor (RF) is calculated for a $1^{\prime \prime}$ conduit having a cable qualification rating of 41 minutes in an area provided with automatic fire suppression having a fire loading of 20,223 BTU/FT2:

FIRE LOAD (FL) CABLE QUAL. RATING (CQR) RESULT $(R F)=C Q R / F L$

15 min .
123 min.
RF=8.2
(20,223 BTU/Ft2)
( $=41$ min.x $3 \mathrm{w} /$ suppression)
NOTE THAT THIS METHOD IS CONSERVATIVE BECAUSE IT PRESUMES COMPLETE COMBUSTION OF ALL COMBUSTIBLES IN THE FIRE AREA OR ZONE OF CONCERN.

## 2.3b Method 2

The second method for evaluating the cable qualification ratings of the ThermoLag fire barriers with respect to the actual fire hazards is by the use of fire modelling.

GPU Nuclear has used EPRI' enhenced FIVE Fire modeling to approximate exposure time-temperature history for Thermo-Lag configurations in five of seven fire zones. The results are contained in the Appendices to this submittal. The following summarizes the EPRI technique and the results:

The source document for this discussion is EPRI Report Project 3385-05 "Methods for Evaluation of Cable Wrap Fire Barrier Performance". The fire hazard tool will calculate location dependent exposure time-tempersture histories, as discussed above, resulting from defined fire scenarios. These scenarios are established by assessing the location and distribution of in-situ and transient hazards in the area under evaluation. Exposures can be determined at any selected location.

Fire growth and propagation are based on actual fire test data. The tool therefore provides estimates of fire exposures for installed raceway fire barrier envelopes including the time required for the fire to grow and propagate and the time to reach critical temperatures at the envelopes. The timing estimated by the tool provides a basis for quantítative assessment of important time dependent measures such as fire brigade response.

The fire modelling method is described in FIVE (EPRI' s Fire Induced Vulnerability Evaluation, TR 100370), Section 10.4, April 1992. The model utilizes the FIVE plume/ceiling jet/hot gas layer correlations to determine pre-flaghover conditions within the evaluated fire zone/area.

Enhancements to the FIVE fire modeliing method, documented in the EPRI "Fire Risk Analysis Implementation Guide" (TR 104030), Draft January, 1994, are incorporated into the fire hazard tool. These enhancements are based on conservative interpretation of actual fire test data and provide the capability of calculating fire growth/propagation in addition to providing improved definition of the burning characteristics of combustible materials typically found in power plants. The fire hazard tool also uses data published by NEI regarding the burning characteristics of Thermo-Lag (NUMARC Thermo-Lag 330-1 Combustibility Evaluation Methodology Plant Screening Guide, September 1993).

The fire hazard tool calculates temperatures at specified time steps in order to define temperature exposure profiles at raceway fire barrier envelope locations. The temperatures at each time step, prior to fire zone/area compartment fiashover, are determined from the FIVE equations considering the enhancements incorporated in the Fire Riak Analysis Implementation Guide. The temperature results are not sensitive to the size of the time step selected. Compartment flashover is conservatively assumed to occur when the hot gas layer reaches 1000 deg $F$. The temperature profile at the time is taken as the ASTM E-119 timetemperature history at all locations within the fire zone/area compartment.

Field wilkdowns are performed to identify and locate all potential fire ignition dources which could potentially impact Thermo-Lag raceway fire barrier envelopes. FIVE Table 1.2 is used as guide in identifying the typee of credible ignition sources.

The approsch described in EPRI's Fire Risk Analysis Implementation Guide (pages 4-18 through 4-27 and Appendix E) is used to screen those ignition sources that do not result in damaging fires and define scenarios with potential for damage. The field walkdowns use pre-calculated critical distance values which are developed using the FIVE fire modelling methods (FIVE Section 10.4).

Ignition sources which are determined to have no potential to cause damage (other than to the ignition source itself) are eliminated from further consideration. Ignition sources which are determined to have the potential of igniting adjacent cable, Thermo-Lag or other combustibles are identified for detailed fire modelling to be performed as described below.

FIVE analyses are performed using computerized Excel spreadsheets similar in format to FIVE worksheets 1,2 and 3 . Time-temperature histories are developed by inputting the heat release rate (HRR) and total heat content of the fuel ( $Q$ Tot) for the fire source into the FIVE equations at the appropriate time steps.

Most material specific parameters pertinent to fire growth and propagation are extracted directly from FIVE. Physical configurations are conservatively determined from the walkdowns. There are conservatisms built into the model. One example of the conservatism of the fire model is the assumption that the fire begins to burn at its peak release rate at timeso. This assumption results in the shortest time to reach peak temperatures (in the plume, ceiling jet and hot gas layer). This is conservative as it taaximizes the incident heat flux and temperatures at the target raceway fire barrier envelope. NOTE ALSO THAT THE MODEL TAKES NO CREDIT FOR THE PRESENCE OF AUTOMATIC SUPPRESSION OR ANY MANUAL FIRE FIGHTING ACTIVITIES.

Implementation of the fire hazard tool is accomplished as folluws:
Equipment is selected for evaluation as a fixed ignition source based on criteria contained in the FIVE methodology guide. Equipment that does not meet the selection criteria may be included for evaluation as an ignition source if it is in close proximity to an important piece of equipment or associated cable.

Fire modelling of fixed ignition sources is performed in e stepwise Fashion ueing the FIVE methodology and Fire Risk Analysis Implementation (FPRA) methodologies developed by EPRI. The detailed steps involved in modelling are provided below. Not all steps of the fire modelling were necessary to identify and evaluate the effect of each ignition source on other cables and equipment. In most cases, the conclusion that the ignition source causes no adverse impact was reached upon completion of the deterministic steps 1 through 9 .

1. Determine the physical characteristics of the fire zone. This requires floor area, ceiling height, room volume, maximum fire zone ambient temperature and identification of any features that could affect the analysis. This information is obtained from plant design documents.
2. Identify all fixed ignition sources in the fire sone. This information is collected from plant databases and verified by walkdown of the zone. During the course of the walkdown, any material etorage loctions are also identified. These locations are treated as fixed ignition sources.
3. Select an appropriate hat lose factor. The heat lose factor accounts for heat absorbed into the fire zone floor ceiling and walls. A figure of 0.94 is utilized in modelling based on guidance in the EPRI Report Project 338505 "Methods for Evaluation of Cable Wrap Fire Barrier Performance". Longer duration scenarios (greater than 5 minutes) such as the formation of a damag-ing hot gas layer, have been experimentally determined to have heat lose factors in the range of 0.94 to 0.98 .
4. Determine target damage and ignition temperatures. For Thermo-Lag, an ignition temperature of 1000 DEG $F$ is used based on guidance in the Thermo-Lag Combustibility Evaluation Methodology Plant Screening Guide.
5. Determine target damage radiant heat fluxes. For Thermo-Lag, an ignition heat flux of 2.2 BTU/E/FT2 is used based upon guidance in the Thermo-Lag Combustibilty Evaluation Methodology Plant Screen Guide.
6. Identify Heat Release Rates (HRRs) for the different categories of fixed ignition sources located within the fire zone. The FPRA manual provides HRRs for most of the standard equipment categories. In cases where HRRs are not explicitly stated, judgement is used to eelect an HRR. For fixed ignition sources containing oil, the HRR is calculated from the amount of oil contained in the source and the potential surface area of the oil spill.
7. Assign a location factor for each fixed ignition source in the fire zone. The location factor accounts for the higher plume temperatures found for fires near walls and in corners.
8. Calculate in-plume damage heights and radiant damage ranges for each fixed ignition source located in the fire zone. This calculation uses equations contained in the FIVE manual.
9. Determine targets for each ignition source. This step utilizes the damage heights and ranges calculated in step 8 . Any piece of equipment, cable, conduit or tray within the damage heights and ranges were noted. Targets that are outside of the radiant damage range and are not located in the plume are evaluated for potential damage from the ceiling jet. Fixed ignition sources without any potential targets are excluded from further analysis other than the loss of the fixed ignition source itself and impact of forming a damaging hot gas layer.

If a target is found to be within the ignition range of aixed ignition source, the target is in turn treated as an ignition source and additional targets are identified. This models the propagation of a fire from a source to a series of combustible targets. Additionally, the heat content of the ignited target must be considered to determine if the formation of a damaging HGL is possible.

Modeling of transient ignition sources and combustibles is similar to fixed ignition sources with some additional steps to account for the uncertainty in location and the amount of combustibles. These additional steps are as follows:
10. Determine what transient combustibles are likely to be present or traverse the fire zone. Due to the potential for large HRRs from oil fires, special emphasis was placed on identifying oil lubricated components as well as any such equipment that was likely to be reached by traversing the fire zone under analysis. The type of container in which oil is transported is also of importance. Oil transported in 55 gallon drums or NFPA approved containers are assumed to be not exposed to potential ignition sources.
11. Select an appropriate transient fuel package based on what types of transient materials are likely to be present in the fire zone. The fuel packages and their asmociated HRRE are selected from Sandia National Laboratory (SNL) testing.
12. Determine target sets based on specific floor locations. Target sets that could be damaged by a fire involving the traneient fuel package are identified.

Once the fire hazard tool has determined the time-temperature history for each scenario, the EPRI "Barrier Performance Tool" is then utilized to establish the exposure to the thermo-lag in each scenario equivalent to an ASTM E-119 exposure fire. Inputs to the tool are the calculated time temperature history as determined by the fire hazard tool.

This tool is based upon the premise that two different exposure histories, which would cause two identical fire barrier raceway configurations to reach the same failure point would have equal areas under their incident hect flux curves.

The "Barrier Performance Tool" therefore calculates the total heat load for the exposure history developed for each scenario by the fire hazard tool.

The duration as expressed by the "Cable qualification Rating" is then compared with the calculated equivalent ASTM E-119 duration for each scenario to assess the calculated severity of each scenario with an ASTM E-119 exposure.

### 3.0 Sumakary of results

The results of applying the above methodology towards establishing an actual fire rating and a cable qualification rating for Thermo-Lag fire barriers are as follows:


The $2^{\prime \prime}$ conduit, $2^{\prime \prime}$ conduit radial bend, $2^{\prime \prime}$ condulet and $2^{\prime \prime}$ penetration elements will be upgraded to provide an "actual" fire rating of one hour and will therefore meet the requirements of III.G.2.C. Materiale which successfully demonstrate a 60 minute fire endurance rating for these conduits (either by themselves or in conjunction with the existing Thermo-Lag installations) will be installed. Such changes will not be submitted for NRC approval but handled under existing licensing conditions; ie. 50.59 process. The commitment is to upgrade these envelopes to an "actual" fire rating of 60 minutes.

All barriers in this fire zone currently will be upgraded to meet the requirements of Appendix R section III.G.2.c. Therefore, no exemptions will be necessary.

| $\begin{aligned} & \text { ENVELOPE } \\ & \text { NO. } \end{aligned}$ | TYPE | No. <br> ELEMENTS | ACTUAL RTG. | CABLE RTG. | NEI TEST QUALIFICATION |
| :---: | :---: | :---: | :---: | :---: | :---: |
| CNCA1041 | 3" Conduit <br> 3" Box (Condulet) <br> 3" Conduit <br> (Penetration) | $\begin{aligned} & 1 \\ & 1 \\ & 1 \end{aligned}$ | $\begin{aligned} & 39 \text { min. } \\ & 39 \text { min. } \end{aligned}$ | $\begin{aligned} & 60 \mathrm{~min} . \\ & 60 \mathrm{~min} . \end{aligned}$ | $\begin{aligned} & 2-1 \\ & 2-1 \end{aligned}$ |
| CNCA1043 | 3" Conduit <br> 3" Box (Condulet) <br> 3" Box Penetration | $\begin{aligned} & 1 \\ & 1 \\ & 1 \end{aligned}$ | $\begin{aligned} & 39 \text { min. } \\ & 39 \text { min. } \end{aligned}$ | $\begin{aligned} & 60 \mathrm{~min} . \\ & 60 \mathrm{~min} . \end{aligned}$ | $\begin{aligned} & 2-1 \\ & 2-1 \end{aligned}$ |
| CNPA1042 | 1" Conduit <br> 1" Box (Condulet) <br> 1" Box Penetration <br> 1" Conduit <br> (Penetration) | $\begin{aligned} & 2 \\ & 1 \\ & 1 \\ & 1 \end{aligned}$ | $\begin{aligned} & 27 \mathrm{~min} . \\ & 27 \mathrm{~min} . \end{aligned}$ | $\begin{aligned} & 41 \text { min. } \\ & 42 \text { min. } \end{aligned}$ | $\begin{aligned} & 2-1 \\ & 2-1 \end{aligned}$ |
| NO. NUMBER | 10'X5'X1' Box | 1 |  |  |  |
| DUCTWORK | $\text { From } \begin{aligned} 40^{\prime \prime} \times 16^{n} \\ 30^{\prime \prime} \times 18^{\prime \prime} \end{aligned}$ | NA | NO | ING |  |

 requirements of III.G.2.c because their "CABLE QUALIPICATION RATING" is at least one hour. The heat transfer model which projects temperatures inside these raceways out to 60 minutes was used to compare raceway temperatures at 60 minutes to the cable insulation failure threshold temperatures for required safe shutdown circuits inside these raceways. For these raceways, the projected raceway temperatures at 60 minutes are below the cable insulation failure threshold temperatures by more than $70 \mathrm{deg} F$; therefore these raceway barriers have a rating of 60 minutes. No additional justification is required for these barriers.

The $1^{\prime \prime}$ conduit, $1^{\prime \prime}$ condulet and $1^{\prime \prime}$ penetration elementg will be upgraded to provide an "actual" fire rating of one hour and will therefore meet the requirements of III.G.2.c. Materials which successfully demonstrate a 60 minute fire endurance rating for these conduits (either by themselves or in conjunction with the existing Thermo-Lag installations) will be installed. Such changes will not be submitted for NRC approval but handled under existing licensing conditions; ie. 50.59 process. The commitment is to upgrade these envelopes to an "actual" fire rating of 60 minutes.

The $10^{\prime} \mathrm{X} 5^{\prime} \mathrm{XI}$ ' box (HOLD)
Based upon existing information, a fire endurance rating cannot be established for the Thermo-Lag installed on the ductwork fire barriers. These barriers provide insulation for ductwork which is part of the ventilation system for the "A" Switchgear Room which is Fire zone $O B-F Z-6 A$. The insulated ductwork is routed through the "B" Switchgear Room or Fire zone OB-FZ-6B. It was provided with the Thermo-Lag for the ductwork in the event of a fire in the "B" Switchgear Room to maintain ventilation in the "A" Switchgear Room. A "rated" configuration for these barriers is not critical to the ability of the ventilation system to maintain an adequate environment in the "A" Switchgear Room. In fact, acceptable duct internal temperatures for a "rated" barrier ( 250 deg $F$ above ambient) are not desirable from a practical standpoint even though a "rated" barrier may technically meet Appendix R. GPU Nuclear does not plan on establishing a fire endurance rating for the Thermo-Lag installed on ductwork. Corrective action will consist of insuring adequate ventilation in fire zone OB-FZ-6A if a fire occurs in fire zone $O B-F 2-6 B$. Adequate ventilation may be insured by operating procedure changes if analysis of the existing ventilation system configuration
with Thermo-Lag serving as duct insulation provides an adequate basis for leaving the existing configuration as is. Preliminary results of our analysis indicates that loss of ventilation to the "A" Switchgear Room results in acceptabla temperatures after one hour for equipment in this room required to operate in the event of a fire in the "B" Switchgear Room. If adequate ventilation cannot be proven analytically, corrective action may take the form of modifications to the " $A$ " Switchgear Room ventilation system.

All electrical raceway barriers in this fire zone currently meet the requirements of Appendix R, section III.G.2.C or will be upgraded to meet the aforementioned requirements. Therefore, no exemptions will be neceseary.

The Thermo-Lag installed on the ductwork in this zone does not require a fire endurance rating. Corrective action will be based upon completion of a ventilation system analysis which considers Thermo-Lag as duct insulation, not a "rated" barrier as stated previously. By insuring edequate ventilation for the "A" Switchgear Room if a fire occurs in the "B" Switchgear Room, either procedurally or by modification, GPU Nuclear will meet the requirements of Section IIIG.1.a in that "one train of systems necessary to achieve and maintain hot shutdown from either the control room or emergency control station(s) is free of fire damage". Completion of the aforementioned ventilation system analysis will identify the scope of work required to insure adequate ventilation for the "A" Switchgear Room.
3.3 Reactor Building 51, Elevation (Fire Ione RB-F8-1D)

| $\begin{aligned} & \text { ENVELOPE } \\ & \text { NO. } \end{aligned}$ | TYPE | NO. ELEMENTS | $\begin{aligned} & \text { ACTUAL } \\ & \text { RTG. } \end{aligned}$ | $\begin{gathered} \text { CABLE } \\ \text { RTG. } \end{gathered}$ | NEI TEST QUALIFICATION |
| :---: | :---: | :---: | :---: | :---: | :---: |
| CGCA1027 | 2" Conduit | 10 | 39 min . | 60 min . | 2-1 |
|  | 2" Radial Bend Conduit | 8 | 39 min . | 60 min . | 2-1 |
|  | 2* Box (Condulet) | 2 | 39 min . | 60 min . | 2-1 |
|  | 2" Conduit <br> (Penetration) | 1 | 39 min . | 60 min . | 2-1 |
|  | 2" Box Penetration | 1 | HOLD |  |  |
| CGPA3026 | 2* Conduit | 10 | 39 min . | 60 min . | 2-1 |
|  | 2" Radial Bend Conduit | 8 | 39 min . | 60 min . | 2-1 |
|  | $2^{\prime \prime}$ Box (Condulet) | 2 | 39 min . | 60 min . | 2-1 |
|  | 2" Conduit <br> (Penetration) | 1 | 39 min . | 60 min . | 2-1 |
|  | 2" Box Penetration | 1 | HOLD |  |  |
| CRCA1026 | 2" Conduit | 4 | 39 min . | 60 min . | 2-1 |
|  | 2" Radial Bend Conduit | 4 | 39 min . | 60 min . | 2-1 |
|  | 2" Conduit <br> (Penetration) | 1 | 39 min . | 60 min . | 2-1 |
|  | 2" Conduit | 1 | HOLD |  |  |

To summarize the above results, $2^{n}$ conduit, $2^{\prime \prime}$ conduit radial bends, $2^{\text {n }}$ condulet and $2^{\prime \prime}$ penetration elements meet the requirements of III.G.2.c because their "CABLE QUALIFICATION RATING" is at least one hour. The heat transfer model which projects temperatures inside these raceways out to 60 minutes was used to compare raceway temperatures at 60 minutes to the cable insulation failure threshold lemperatures for required safe shutdown circuits inside these raceways. For
these raceways, the projected raceway temperatures at 60 minutes are below the cable insulation failure threshold temperatures by more than 70 deg $F$; therefore these raceway barriers have a rating of 60 minutes. No additional fustification is required for these barriers.

| $\begin{aligned} & \text { ENVELOPE } \\ & \text { NO. } \end{aligned}$ | TYPE | NO. ELEMENTS | $\begin{aligned} & \text { ACTUAL } \\ & \text { RTG. } \end{aligned}$ | $\begin{gathered} \text { CABLE } \\ \text { RTG. } \end{gathered}$ | NEI TEST QUALIFICAIION |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 62-153 | 2" Conduit | 4 | 39 min . | 56 min . | 2-1 |
|  | 2" Radial Bend | 4 | 39 min . | 56 min . | 2-1 |
|  | Conduit |  |  |  |  |
|  | 2" Conduit | 1 | 39 min . | 56 min . | 2-1 |
|  |  |  | HOLD |  |  |
|  | 2" Conduit <br> (Penetration) | 1 |  |  |  |
| CGCR2086 | 1" Conduit | 3 | 27 min . | 36 min . | 2-1 |
|  | 1" Radial Bend | 3 | 27 min . | 36 min . | 2-1 |
|  | Conduit |  |  |  |  |
|  | $1^{\text {n }}$ Box (Condulet) | 1 | 27 min . | 36 min . | 2-1 |
|  | 1" Conduit <br> (Penetration) | 2 | 27 min . | 36 min . | 2-1 |
| CGCR3021 | 1" Conduit | 9 | 27 min . | 41 min . | 2-1 |
|  | 1" Radial Bend Conduit | 7 | 27 min . | 41 min . | 2-1 |
|  | 1 " Box (Condulet) | 3 | 27 min . | 41 min . | 2-1 |
|  | 1" Conduit | 2 | 27 min . | 41 min . | 2-1 |

The above three envelopes (62-153, CGCR2086, CGCR3021) will be the subject of an exemption request from the requirement in Appendix R, Section III.G.2.C, for a one hour rated fire barrier in fire zone RB-FZ-1D. The heat transfer model which projects temperatures inside these raceways out to 60 minutes was compared with the cable insulation failure threshold temperatures for required safe shutdown circuits inside these raceways. For these raceways, the projected raceway temperatures at the duration listed are below the cable insulation failure threshold temperatures by more than 70 deg $F$; thereffre thise raceway barriers have a cable qualification rating as listed stuve.

| ENVELOPE NO. | TYPE | NO. ELEMENTS | $\begin{aligned} & \text { ACTUAL } \\ & \text { RTG. } \end{aligned}$ | $\begin{aligned} & \text { CABLE } \\ & \text { RTG. } \end{aligned}$ | ```NEI TEST QUALIFICATION``` |
| :---: | :---: | :---: | :---: | :---: | :---: |
| CGXA3028 | 1" Conduit | 8 | 27 min . | 14 min . | 2-1 |
|  | 1" Radial Bend | 13 | 27 min . | 14 min . | 2-1 |
|  | 1" Box (Condulet) | 2 | 27 min . | 14 min . | 2-1 |
|  | 1" Conduit <br> (Penetration) | 1 | 27 min . | 14 min . | 2-1 |
|  | 1 " Box Penetration | 1 |  |  |  |

The 1 " conduit, 1 inch conduit radial bends, 1 inch condulet, and 1 inch conduit penetration elements will be upgraded to provide an "actual" fire rating of one hour and will therefore meet the requirements of III.G.2.c. Materials which successfully demonstrate a 60 minute fire endurance rating for these conduits (either by themselves or in conjunction with the existing Thermo-Lag installations) will be installed. Such changes will not be submitted for NRC approval but handled under existing licensing conditions; ie. 50.59 process. The commitment is to upgrade these envelopes to an "actual" fire rating of 60 minutes.

| $\begin{aligned} & \text { ENVELOPE } \\ & \text { NO. } \end{aligned}$ | TYPE | $\begin{aligned} & \text { NO. } \\ & \text { ELEMENTS } \end{aligned}$ | $\begin{gathered} \text { ACTUAL } \\ \text { RTG. } \end{gathered}$ | $\begin{gathered} \text { CABLE } \\ \text { RTG. } \end{gathered}$ | NEI TEST QUALIFICATION |
| :---: | :---: | :---: | :---: | :---: | :---: |
| CGPA2008 | $3^{n}$ Conduit | 2 | 39 min . | 60 min . | 2-1 |
|  | $3^{\prime \prime}$ Box (Condulet) | 3 | 39 min . | 60 min . | 2-1 |
|  | $3^{n}$ Conduit | 1 | 39 min . | 60 min . | 2-1 |
|  | (Penetration) <br> 3" Conduit <br> (Penetration) | 1 |  |  |  |
| $\begin{aligned} & \text { ENVELOPE } \\ & \text { NO. } \end{aligned}$ | TYPE | NO. ELEMENTS | $\begin{gathered} \text { ACTUAL } \\ \text { RTG. } \end{gathered}$ | $\begin{gathered} \text { CABLE } \\ \text { RTG. } \end{gathered}$ | NEI TEST QUALIFICATION |
| CRCA1026 | 2" Conduit | 1 | 39 min . | 60 min . | 2-1 |
|  | 2" Radial Bend | 3 | 39 min . | 60 min . | 2-1 |
|  | Conduit |  |  |  |  |
|  | 2" Box (Condulet) | 1 | 39 min . | 60 min . | 2-1 |
|  | 2" Conduit | 2 |  |  |  |

To summarize the above results, $2^{\prime \prime}$ and $3^{\prime \prime}$ conduit, $2^{\prime \prime}$ conduit radial bends and $2^{\prime \prime}$ and $3^{\prime \prime}$ condulet elements meet the requirements of III.G.2.c because their "CABLE QUALIFICATION RATING" is at least one hour. The heat transfer model which projects temperatures inside these raceways out to 60 minutes was used to compare raceway temperatures at 60 minutes to the cable insulation failure threshold temperatures for required safe shutdown circuits inside these raceways. For these raceways, the projected raceway temperatures at 60 minutes are below the cable insulation failure threshold temperstures by more than 70 deg $F$; therefore these raceway barriers have a rating of 60 minutes. No additional justification is required for these barriers.

| $\begin{aligned} & \text { ENVELOPE } \\ & \text { NO. } \end{aligned}$ | TYPE | NO. ELEMENTS | $\begin{aligned} & \text { ACTUAL } \\ & \text { RTG. } \end{aligned}$ | $\begin{aligned} & \text { CABLE } \\ & \text { RTG. } \end{aligned}$ | NEI TEST QUALIFICATION |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 62-153 | 2" Conduit | 1 | 39 min . | 56 min . | 2-1 |
|  | 2" Radial Bend | 3 | 39 min . | 56 min . | 2-1 |
|  | Conduit |  |  |  |  |
|  | 2" Box (Condulet) | 1 | 39 min . | 56 min . | 2-1 |
|  | 2" Conduit <br> (Penetration) | 2 | HOLD |  |  |
| CGCA2010 | 1.5" Conduit | 2 | 27 min . | 36 min . | 2-1 |
|  | 1.5" Box (Condulet) | 3 | 27 min . | 36 min . | 2-1 |
|  | 1. 5" Conduit <br> (Penetration) | 1 | 27 min . | 36 min . | 2-1 |
|  | 1.5" Conduit <br> (Penetration) | 1 | HOLD |  |  |
| CGCR3021 | 1 " Conduit | 10 | 27 min . | 42 min . | 2-1 |
|  | 1" Radial Bend Conduit | 9 | 27 min . | 41 min . | 2-1 |
|  | 1 " Box (Condulet.) | 3 | 27 min . | 41 min . | 2-1 |
|  | 1" Conduit <br> (Penetration) | 1 | 27 min . | 41 min . | 2-1 |
|  | Drywell Pen. Box | 1 | HOLD |  |  |
| PEN | Drywell Pen. Box | 4 |  |  |  |

The above envelopes (62-153, CGCA2010, CGCR3021) will be the subject of an exemption request from the requirement in Appendix R, section III.G.2.C, for a one hour rated fire barrier in fire zone RB-FZ-1E. The heat transfer model which projects temperatures inside these raceways out to 60 minutes was compared with the cable insulation failure threshold temperatures for reguired safe shutdown circuits inside these raceways. For these raceways, the projected raceway temperatures at the duration lieted are below the cable insulation failure threshold temperatures by more than 70 deg $F$; therefore these raceway barriers have a cable qualification rating as listed above.

| ENVELOPE <br> NO. | TYPE | NO. <br> ELEMENTS | ACTUAL <br> RTG. | CABLE <br> RTG. | NEI TEST <br> QURLIFICATION |
| :--- | :--- | :--- | :--- | :--- | :--- |
| CGXR2051 | $1^{\prime \prime}$ Conduit |  | 3 | 27 min. | 23 min. |

The $1^{\prime \prime}$ conduit, 1 inch conduit radial bends, 1 inch condulet, and 1 inch conduit penetration elements will be upgraded to provide an "actual" fire rating of one hour and will therefore meet the requirements of III.G.2.c. Materials which successfully demonstrate a 60 minute fire endurance rating for these conduits (either by themselves or in conjunction with the existing Thermo-Lag installations) will be installed. Such changes will not be submitted for NRC approval but handled under existing licensing conditions; ie. 50.59 process. The commitment is to upgrade these envelopes to an "actual" fire rating of 60 minutes.

### 3.5 Reactor Building -19' Elevation (Fire zone RB-FZ-1F2)

| $\begin{aligned} & \text { ENVELOPE } \\ & \text { NO. } \end{aligned}$ | TYPE | $\begin{gathered} \text { NO. } \\ \text { ELEMENTS } \end{gathered}$ | $\begin{aligned} & \text { ACTUAL } \\ & \text { RTG. } \end{aligned}$ | $\begin{gathered} \text { CABLE } \\ \text { RTG. } \end{gathered}$ | NEI TEST QUALIFICATION |
| :---: | :---: | :---: | :---: | :---: | :---: |
| CGPA2002 | 1.5" Conduit | 2 | 27 min . | 41 min . | 2-1 |
|  | 1.5" Radial Bend Conduit | 1 | 27 min . | 41 min . | 2-1 |
|  | 1.5" Box (Condulet) | 2 | 27 min . | 41 min . | 2-1 |
|  | 1.5" Conduit <br> (Penetration) | 1 | 27 min . | 41 min . | 2-1 |
|  | 1. 5" Box Penetration | n 1 |  |  |  |

The above envelope (CGPA2002) will be the subject of an exemption request from the requirement in Appendix R, Section III.G.2.c, for a one hour rated fire barrier in fire zone RB-FZ-1F2. The heat transfer model which projects temperatures inside these raceways out to 60 minutes was compared with the cable insulation failure threshold temperatures for required safe shutdown circuits inside these raceways. For these raceways, the projected raceway temperaturea at the duration listed are below the cable insulation failure threshold temperatures by more than 70 deg $F$; therefore these raceway barriers have a cable qualification rating as listed above.


| $\begin{aligned} & \text { ENVELOPE } \\ & \text { NO. } \end{aligned}$ | TYPE E | NO. ELEMENTS | $\begin{gathered} \text { ACTUAL } \\ \text { RTG. } \end{gathered}$ | $\begin{gathered} \text { CABLE } \\ \text { RTG. } \end{gathered}$ | NEI TEST QUALIFICATION |
| :---: | :---: | :---: | :---: | :---: | :---: |
| CGPTB020 | 1.5* Conduit | 12 | 27 min . | 36 min . | 2-1 |
|  | 1.5" Radial Bend conduit | 12 | 27 min . | 36 min . | 2-1 |
|  | 1.5" Box (Condulet) | 3 | 27 min . | 36 min . | 2-1 |
|  | 1.5" Conduit <br> (Penetration) | 1 | 27 min . | 36 min . | 2-1 |
|  | 1.5" Junction Box | 2 |  |  |  |
| CGPTB029 | 1" Conduit | 9 | 27 min . | 36 min . | 2-1 |
|  | 1" Radial Bend Conduit | 7 | 27 min . | 36 min . | 2-1 |
|  | 1" Box (Condulet) 6"x6" Junction Box | $\begin{aligned} & 1 \\ & 1 \end{aligned}$ | 27 min . | 36 min . | 2-1 |
| CGPTB030 | 1.5" Conduit | 11 | 27 min . | 36 min . | 2-1 |
|  | 1.5" Radial Bend Conduit | 7 | 27 min . | 36 min . | 2-1 |
|  | 1.5" Box (Condulet) | 2 | 27 min . | 36 min . | 2-1 |

The above envelopes $(14-25,14-28,62-93,86-71$, CGCTB017, CGCTBO71, CGPTBO20, CGPTBO29 and CGPTBO30) will be the subject of an exemption request from the requirement in Appendix R, Section III.G.2.C, for a one hour rated fire barrier in fire zone TB-FZ-11D. The heat transfer model which projects temperatures inside these raceways out to 60 minutes was compared with the cable insulation failure threshold temperatures for required safe shutdown circuits inside these raceways. For these raceways, the projected raceway temperatures at the duration listed are below the cable insulation failure threshold temperatures by more than 70 deg $F$; therefore these raceway barriers have a cable qualification rating as listed above.

| ENVELOPE <br> NO. | TYPE | NO. <br> ELEMENTS | ACTUAL <br> RTG. | CABLE | RTG. |
| :---: | :---: | :---: | :---: | :---: | :---: | | NEI TEST |
| :---: |
| QUALIFICATION |

The $1.5^{\prime \prime}$ flexible conduit will be upgraded to provide an "actual" fire rating of one hour and will therefore meet the requirements of III.G.2.C. GPU Nuclear is currently planning on designing and installing the upgrade for these elements to conform to Texas Utilities Scheme $11-2$ for a $1-1 / 2^{\prime \prime}$ airdrop. These tests were successful in establishing a 60 minute fire endurance rating. This constitutes GPU Nuclear's current design detail for the aforementioned upgrades. If other types of upgrades which provide an "actual" fire rating of 60 minutes are identified due to more industry testing, an alternative configuration could be used. Such changes will not be submitted for NRC approval but handled under existing licensing conditions; ie. 50.59 process. The commitment is to upgrade these envelopes to an "actual" fire rating of 60 minutes.

### 3.7 Turbine Building, Switchgear Room, West End, Mezzanine (Fire Zone TB-FZ-11C)

| $\begin{aligned} & \text { ENVELOPE } \\ & \text { NO. } \end{aligned}$ | TYPE | NO, ELEMENTS | ACTUAL RTG. | CABLE RTG. | $\begin{gathered} \text { NEI TEST } \\ \text { QUALIFICATION } \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 86-71 | $4^{n}$ Conduit | 2 | 91 min . | 102 min . | 2-3 |
|  | $4^{\prime \prime}$ Radial Bend | 4 | 91 min . | 102 min . | 2-3 |
|  | 4" Conduit | 4 |  |  |  |

The above envelope (86-71) will be the subject of an exemption request from the requirement in Appendix R, Section III.G.2.a, for a three hour rated fire barrier in fire zone TB-FZ-11C. Request an exemption from the requirement for an area wide automatic suppression system. The temperature inside these raceways at 102 minutes, when the test was terminated, was compared with the cable insulation failure threshold temperatures for required safe shutdown circuits inside these raceways. For these raceways, the projected raceway temperatures at the duration listed are below the cable insulation failure threshold temperatures by more than 70 deg $F$; therefore these raceway barriers have a cable qualification rating as listed above.

### 4.0 REFERENCES

4.1 NRC Generic Letter 86-10, Supplement 1, Enclosure 1, "FIRE ENDURANCE TEST ACCEPTANCE CRITERIA FOR FIRE BARRIER SYSTEMS USED TO SEPARATE REDUNDANT SAFE SHUTDOWN TRAINS WITHIN THE SAME FIRE AREA", dated March 25, 1994.
4.2 10 CFR Part 50 APpendix R, "FIRE PRCTECTION PROGRAM FOR NUCLEAR POWER FACILITIES OPERATING PRIOR TO JANUARY 1, 1979*.

4:3 NEI Report No. 0784-00001-TR-02, Revision 1, "NEI APPLICATION GUIDE FOR EVALUATION OF THERMO-LAG 330 FIRE BARRIER SYSTEMS*.
4.4 GPU Nuclear Oyster Creek Nuclear Generating Station Fire Hazards Analysis Report (FHAR) No. 990-1746, Revision 7.
4.5 NFPA 251 (ASTM E-119), "STANDARD FIRE TESTS OF BUILDING CONSTRUCTION AND MATERIALS".
4.6 Sandia Test Report SAND90-0696 May, 1991 "AN INVESTIGATION OF THE EFFECTS OF THERMAL AGING ON THE FIRE DAMAGEABILITY OF ELECTRICAL CABLES".
4.7 Gilbert Commonwealth Letter G/C/TMI-1CS/16503 Sept. 15,1988, J. Brendlen to J.W.Langenbach, "TSI DERATING CHECK".
4.8 NFPA Fire Protection Handbook, Seventeenth Edition.
4.9 EPRI TR 100370, April, 1992, "FIRE INDUCED VULNERABILITY EVALUATION".
4. 10 EPRI TR 104030, draft January, 1994, "FIRE RISK ANALYSIS IMPLEMENTATION GUIDE".
4. 11 NUMARC, SEPt 1993, "THERMO-LAG 330-1 COMBUSTIBILITY EVALUATION METHODOLOGY PLANT SCREENING GUIDE".
4. 12 EPRI Project 3385-05, June 1995, "METHODS FOR EVALUATION OF CABLE WRAP FIRE BARRIER PERFORMANCE".

## APPENDICES

A. USING FIVE FIRE MODELLING TO APPROXIMATE EXPOSURE TIME-HISTORY FOR THERMO-LAG CONFIGURATIONS IN OC FIRE ZONE RB-FZ-1D.
B. USING FIVE FIRE MODELLING TO APPROXIMATE EXPOSURE TIME-HISTORY FOR THERMO-LAG CONFIGURATIONS IN OC FIRE ZONE RB-FZ-1E.
C. USING FIVE FIRE MODELLING TO APPROXIMATE EXPOSURE TIME-HISTORY FOR THERMO-LAG CONFIGURATIONS IN OC FIRE ZONE RB-FZ-1F2.
D. USING FIVE FIRE MODELLING TO APPROXIMATE EXPOSURE TIME-HISTORY FOR THERMO-LAG CONFIGURATIONS IN OC FIRE ZONE TB-FZ-11D.
E. USING FIVE FIRE MODELLING TO APPROXIMATE EXPOSURE TIME-HISTORY FOR THERMO-LAG CONFIGURATIONS IN OC FIRE ZONE TB-FZ-11C.

## APPENDIXA

# Using FIVE Fire Modeling to Approximate Exposure TimeHistory for Thermo-Lag Configurations in OC Fire Zone RB-FZ-1D 

Letter Report<br>September 27, 1995

Prepared by: GFUN Risk Analysis Group
Using FIVE Fire Modeling to Approximate Exposure Time-History for Thermo-Lag Configurations

## 1. PURPOSE

The purpose of this report is to document the detailed fire modeling performed for fire scenarios involving Thermo-Lag wrapped fire barriers, as identified during plant walkdowns of OC fire zone RB-FZ-1D. Thermo-Lag exposure time-history results were obtained for each T-L configuration determined to be impacted by a fire source/combustible.

## 2. APPROACH

The approach followed for modeling of fire scenarios is based on methodology detailed in the EPRI "Methods for Evaluation of Cable Wrap Fire Barrier Performance" document (Ref. 1) which was based on fire modeling techniques developed for FIVE (Ref. 2). Fire modeling was accomplished in two steps:

1. Screening walkdowns to eliminate from further consideration those fire ignition sources and potential fire scenarios that cannot develop into damaging fires.
2. Fire modeling of specific scenarios that were not screened out in the first step.

For each Thermo-Lag configuration analyzed, one or more fire scenarios (including bounding scenarios) were postulated based on the following:
a. information gathered during plant walkdowns (i.e., fixed or transient ignition source and Thermo-Lag target location, type and quantity of combustibles present in zone);
b. information obtaired from reviewing applicable sections in Appendix $R$ documents for OC (Ref. 3);
c. information obtained from reviewing OC procedure 120.5, Rev. 5, "Control of Combustibles." (Ref. 4)
d. information provided by GPUN enyineers (specializing in fire protection, risk assessment, mechanical).

Thermo-Lag FIVE analyses were performed using computerized Excel spreadsheets which duplicate FIVE Worksheets i, 2, or 3 and are linked to tables containing the same information as in FIVE Reference Tables. For each Thermo-Lag fire modeling scenario, results are presented as exposure time-temperature profiles. In all scenarios evaluated, time-dependent temperatures were derived by incrementally adding the total heat content of the fuel into the fire compartment hot gas layer.

## 3. FIRE MODELING ASSUMPTIONS

The following Thermo-Lag fire modeling assumptions were made:

1. Worst-case fixed fire source scenarios were modeled for each Thermo-Lag sub-
configuration. The determination of which fixed fire source might cause the worstcase Thermo-Lag fire scenario was made during the plant fire area walkdown and was based on minimum damage threshold heights and distances calculated for typical fixed fire sources (e.g., electrical cabinets, electrical motors, tube oil in pumps) found at Oyster Creek.
2. Worst-case transient fire source scenarios observed during walkdowns were also modeled. Worst-case transient fire scenarios considered were similar to the type anc quantities of transient combustibles allowed by OC station procedure (Ret. 4).
3. To ensure conservative results, worst-case fire plume (or celling jet) scenarios and/or radiant fiux scenario were first modeled. For T-L sub-configurations not in the plume or celling jet, hot gas layer fire scenarios were evaluated.
4. Electrical cable inside cabinets, motor control centers, switchgear and cable trays was assumed to be non-qualified (i.e., non IEEE-383 type). This assumption is conservative since slectrical cable purchased and installed at Oyster Creek in the past decade was IEEE-383 type.
5. The Fire Hazard Tool developed for the EPRI Cable Wrap Fire Barrier Tallored Collaboration assumes that the fire compartment temperature will follow the ASTM E-119 time-temperature curve as soon as the compartment hot gas layer (HGL) reaches $1000^{\circ} \mathrm{F}$. When the HGL temperature reaches $1000^{\circ} \mathrm{F}$, the plume (or celling jet) temperatures are assumed to follow that of the E-119 (see discussion in Appendix B).
6. The Thermo-Lag wrap is assumed to burn by itseff (i.e., even after the initial fire source is out of fuel) unless otherwise stated. Per, the EPRI Method (Ref. 1, Appendix $1-D$ ), credit may be taken for a time lag before the Thermo-Lag ignites based on exposure temperature.
7. The maximum plume and ceiling jet temperature is assumed to be $1600^{\circ} \mathrm{F}$, which corresponds to flame temperature. This assumption is valid until hot gas layer temperature exceeds $1600^{\circ} \mathrm{F}$, at which point the plume temperature is assumed to be equal to hot gas layer temperature.
8. Fire Zone RB-FZ-1D specific data:

- total area of 9,100 ft2 (Ref. 3-Oyster Creek Fire Hazard Analysis Report, Rev. No. 8);
- fire zone ceiling height of 24 feet (Oyster Creek General arrangement Drawings);
- ambient temperature of $100^{\circ} \mathrm{F}$.


## 4. FIRE MODELING RESULTS

### 4.1 Fire Zone RB-FZ-1D

Fire zone RB-FZ-1D is the Reactor Building 51' Elevation. There are three areas in this fire zone that contain the T-L wrap. They are designated as location Nos. 6, 7, and 8 on the general arrangement drawing shown in Appendix D .

## Location No. 6

Four scenarios were evaluated for this T-L configuration. The first two scenarios model autoignition of non-qualified cable trays and their impact on the T-L wrapped condults above. The third scenario evaluates a transient ignition source/combustible that was noted during the walk-downs in the area, and the last scenario models the HGL contribution due to a core spray booster pump oil fire.

Scenario \#1. There are two vertical T-L. wrapped conduit runs that start from Location No. 9 at Elevation 33 ' below and go up to elevation 51' and then curve along the wall. An I\&C cable runs by the two vertical T-L conduits in this area, however, this is a qualified cable and autoignition is not a concern. There are, however, three very loosely-packed cable trays in this area that run about 7 ft . below The T-L wrap. This was modeled here as one fully packed cable tray self-igniting that could impact the T-L wrap. Two ft. of a $24^{\prime \prime}$-wide cable tray was assumed to initially sell-ignite and the T-L target was modeled "in-plume" with the fire source located against the wall. This fire is assumed to propagate horizontally along the exposed cable tray with a fiame speed of 10 feet per hour (per Ref. 1, Appendix I-C).

The heat release rate considered for this scenario (before the T-L wrap is ignited) is as follows:

1. Cable tray - 23.4 BTU/sec- ft? (Ref. 2, FIVE Table 1E, cable 5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); since 4 ft2 are inkially burning, the HRR for this source alone is 93.6 BTU/sec; this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be 141 BTU/s, which includes the HRR from 2 ft 2 of additional cable tray);

The total available heat (Otot) from this cable tray fire source is:

1. Cable tray $-4,000,000$ BTUs [it assumes 10,000 BTU/lbm of cable insulation (per Ref. 6, page E2-1) $\times 10$ linear feet $\times 40 \mathrm{lbm} / \mathrm{linear}$ foot (engineering judgment for $100 \%$ filled, 24 inch wide cable tray)];

## Results

The analysis as shown in Table 1 indicates that the "in-plume" temperatures exceed $1000^{\circ} \mathrm{F}$ at $t=24$ minutes into this scenario, so the T-L wrap will be ignited and burn. However, the HGL temperature is only $112^{\circ} \mathrm{F}$. Once the cable-trays' fire ignites the T-L wrapped conduits, they will start to burn. T-L wrap HRR is assumed to be 8.8 BTU/sec-ft2 (per Ref. 5); the amount of Thermo-Lag that could be involved in this fire scenario is assumed to be 12 linear feet (6 linear feet per conduit). The total surface area is 15.7 th2 (conduits are 4 - inch in diameter with $1 / 2$ inch thick T-L wrap); the HRR is thus $138 \mathrm{BTU} / \mathrm{sec}$. This HRR will be considered only for calculating the HGL temperature rise. The total avallable heat (Ototal) from this amount of T-L fire source is $705,600 \mathrm{BTU}$ (it assumes $58,800 \mathrm{BTU}$ /inear foot, from Rel. 5). A sample model worksheet is provided in Appendix A, Figure A-1.

Table 1. Temperature-Time History for Fire Scenario 1, Location No. 6.

| Time | HRR to | HRIR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL |  |  |  |
| (min.) | (Btu/s) | (Btu/s) | (Btu) | Target - (F) | HGL-(F) |
| 0 | 94 | 94 | 0 | 671 | 100 |
| 6 | 141 | 141 | 50760 | 672 | 102 |
| 12 | 188 | 188 | 118440 | 795 | 104 |
| 18 | 235 | 235 | 203040 | 909 | 107 |
| 24 | 282 | 420 | 354240 | 1018 | 112 |
| 30 | 329 | 467 | 522360 | 1121 | 117 |
| 36 | 376 | 514 | 707400 | 1221 | 123 |
| 42 | 423 | 561 | 909360 | 1317 | 130 |
| 48 | 470 | 608 | 1128240 | 1411 | 138 |
| 54 | 517 | 655 | 1364040 | 1584 | 146 |
| 60 | 564 | 702 | 1616760 | 1593 | 155 |

Scenario \#2. There are three T-L wrapped conduit runs along the wall in this area with two $24^{\text {" }}$-wide cable trays ( 1 ft . apart on top of each other) about 8 ft . directly underneath the T-L conduits. The cable trays are 6 ft . off the floor and are perpendicular to the wall. At the location where this scenario is postulated, this area communicates with a higher elevation through a large opening but there is a 3.5-4' lip on the ceiling and the T-L ends at the ceiling of this lip. Therefore, this area was modeled as a $20^{\prime}$ by $20^{\prime}$ room with a $24^{\prime}$ ceiling for this scenario.

This fire scenario modeled a fixed ignition source/combustible, a 2 foot wide cable tray with non IEEE-383 cable, 6 feet off the room's floor. Since the cable is non-qualified, the cable fire can self-ignite (per Ret. 2, Reference Table 1.2, "Fire Ignition Sources and Frequencies by Applicable Plant Locations) and cable would start to burn. The fire will then propagate (almost immediately) to a second cable tray, located 1 foot above. In addition to the vertical propagation, the fire is also assumed to propagate horizontally along the exposed cable tray with a flame speed of 10 feet per hour (per Ret. 1, Appendix I-C). The targets are three 3-inch diameter conduits wrapped in 0.5 inch thick Thermo-Lag. The T-L. wrapped conduits pass perpendicularly over the cable trays, 8 feet above them. Due to the close distance between the cable trays and T-L wrapped conduits, the T-L will be ignited by the cable tray fire. Once this happens, additional heat is released into this fire zone.

The heat release rates considered for this scenario are as follows (by fire source):

Equation 2 on FIVE manual, page 10.4-10); since 4 ft2 are initially burning, the HRR for this source alone is 93.6 BTU/sec; this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be 140.4 BTU/s, which includes the HRR from 2 ft 2 of additional cable tray);

- second cable tray - 23.4 BTU/sec- ft2 (Ref. 2, FIVE Table 1E, cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); just as with the first cable tray, the HRR for this source alone is 140.4 BTU/sec (since 6 tit is initially assurned to be burning); this HRR will increase with time, as fire propagates horizontally along the tray (l.e., 6 minutes into the fire, the HRR for this source will be 187.2 BTU/s, which includes the HRR from 2 ft 2 of additional cable tray);
- T-L wrap HRR is assumed to be 8.8 BTU/sec-ft2 (per Ref. 5); the amount of Thermo-Lag that could be involved in this fire scenario is assumed to be 7 linear feet (visual observation) for a total surface area of 7.3 ft ; the HRR is thus 64 BTU/sec; this HRR will be considered only for calculating the HGL temperature rise.

The total available heat (Qtot) from each fire source is:

- first cable tray $-4,000,000$ BTUs [it assumes $10,000 \mathrm{BTU} / \mathrm{lbm}$ of cable insulation (per Ref. 6, page E2-1) $\times 10$ linear feet $\times 40$ lom/linear foot (engineering judgment for $100 \%$ filled, 24 inch wide cable tray)];
- second cable tray $-4,000,000$ BTUs [it assumes 10,000 BTU/lbm of cable insulation (per Ref. 6, page E2-1) $\times 10$ linear feet $\times 40 \mathrm{lbm} / \mathrm{linear}$ foot (engineering judgment for $100 \%$ filled, 24 inch wide cable tray)];
- Thermo-Lag wrap - 333,200 BTUs (it assumes $47,600 \mathrm{BTU} / \mathrm{linear}$ foot, from Ref. 5).


## Results

The results are presented in Table 2 below. As indicated by the temperature-time histories in the table, the 'in-plume' temperature at the T-L targets 8 feet above the burning cable trays exceeds $1000^{\circ} \mathrm{F}$ at $\mathrm{t}=12$ minutes, so the T-L wrap will be ignited and burn. However, the HGL temperature is only $263^{\circ} \mathrm{F}$. As indicated by the temperature-time histories in Table 2, postflashover conditions (i.e., above $1000^{\circ}$ F HGL temperature) occur af $t=36$ minutes and the HGL temperatures will follow those of the ASTM E-119 from then on. Appendix B contains the basis for this assumption. A sample model worksheet is provided in Appendix A, Figure A-2.

Table 2. Temperature-Time History for Fire Scenario 2, Location No. 6.

| Time | HRR to | HRR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL |  |  |  |
| (min.) | (Btu/s) | (Bxu/s) | (Btu) | Target-(F) | HGL-(F) |
| 0 | 234 | 234 | 0 | 902 | 100 |
| 6 | 328 | 328 | 118080 | 963 | 161 |
| 12 | 422 | 486 | 293040 | 1212 | 263 |
| 18 | 516 | 580 | 501840 | 1493 | 408 |
| 24 | 610 | 674 | 744480 | 1600 | 612 |
| 30 | 704 | 768 | 1020960 | 1600 | 905 |
| 36 | 798 | 862 | 1331280 | 1600 | 1329 |
| 42 | 892 | 956 | 1675440 | 1600 | 1399\% |
| 48 | 986 | 1050 | 2053440 | 1600 | $1462^{*}$ |
| 54 | 1080 | 1744 | 2465280 | 1600 | $1526^{*}$ |
| 60 | 1774 | 1238 | 2910960 | 1600 | $1550^{*}$ |

Temperatures follow those of ASTM E-119.

Scenario \#3. A wooden ladder was noted stored on the floor during walkdowns of the same area described above for Scenario \#2 for this fire zone. This is modeled here as a transient ignition source/combustible. There is a $24^{n}$-wide cable tray four feet above "in-plumen of thfire which will be ignited due to close proximity to the ladder fire source. It is assumed that 2 ft . of this cable tray will be initially ignited and this fire is assumed to propagate horizontally along the exposed cable tray with a flame speed of 10 feet per hour (per Ref. 1, Appendix 1 C). The HGL temperatures were calculated here for the impact on the T-L in the area 14 ft . above the floor.

The heat release rates considered for this scenario are as follows (by fire source):

- wooden ladder - 12 BTU/sec- ft2 (Ref. 2, FIVE Table 2E); it is assumed that a total surface area of 10 tt 2 are initially burning, the HRR for this source alone is $120 \mathrm{BTU} / \mathrm{sec}$.
- cable tray - 23.4 BTU/sec- ft2 (Ref. 2, FIVE Table 1E, cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); the HRR for this source is 93.6 ETU/sec (since 4 ft 2 is initially assumed to be burning); this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be $140.4 \mathrm{BTU} / \mathrm{s}$, which includes the HRR from 2 ft 2 of additional cable tray);

The total available heat (Qtot) from each fire source is:

- wooden ladder - 400,000 BTUs [it assumes the ladder weighs $50 \mathrm{lbm} \times 8,000 \mathrm{BTU} / \mathrm{bm}$ (per Ref. 6, Page E2-3)];
- cable tray - 4,000,000 BTUs [it assumes 10,000 BTU/lbm of cable insulation (per Ref. 6, page E2-1) $\times 10$ linear feet $\times 40 \mathrm{lbm} / \mathrm{linear}$ foot (engineering judgment for $100 \%$ filled, 24 inch wide cable tray)];


## Resuits

The analysis indicates that as a result of the ladder igniting, the "in-plume" temperatures at the cable tray 4 ft . above reaches $937^{\circ} \mathrm{F}$ at $\mathrm{t}=6$ minutes. Two feet of this cable tray is assumed to ignite at this time and propagate horizontally along the exposed cable tray. The HGL temperatures calculated for the T-L. wrapped conduits 14 ft . above the floor in this area are shown in Table 3 below. As can be seen, the HGL temperatures exceed $1000^{\circ} \mathrm{F}$ at the end of this scenario $(t=60$ minutes $)$. This fire scenario is ended at 1 hour since the $T$-L modeled is assumed to be a 1 -hour wrap. It should also be noted that these HGL temperatures are bounded by Scenario \# 2 above. A sample model worksheet is provided in Appendix A, Figure A-3. The target temperature reflected on this worksheet is the temperature seen by the cable tray 4 feet above the ladder and The T-L wrap ( 14 ft above) sees the HGL temperatures in this scenario.

Table 3. Temperature-Tirne History for Firs Scenario 3, Location No. 6.

| Time | HRR to | HRR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target <br> $(B t u / s)$ | HGL |  |  |  |
| (Btu/s) | (Btu) | Target - (F) | HGL-(F) |  |  |
| 0 | 120 | 120 | 0 | 100 | 100 |
| 6 | 214 | 214 | 77040 | 129 | 129 |
| 12 | 261 | 261 | 171000 | 166 | 166 |
| 18 | 308 | 308 | 281880 | 213 | 213 |
| 24 | 355 | 355 | 409680 | 272 | 272 |
| 30 | 402 | 402 | 554400 | 345 | 345 |
| 36 | 449 | 449 | 716040 | 435 | 435 |
| 42 | 496 | 496 | 894600 | 545 | 545 |
| 48 | 543 | 543 | 1090080 | 683 | 683 |
| 54 | 590 | 590 | 1302480 | 853 | 853 |
| 60 | 637 | 637 | 1531800 | 1066 | 1066 |

Scenario \#4. The fixed ignition source modeled here is the core spray booster pump P-202A. This pump contains up to 2 gailons of lube oil (Ref. 7). Total Q available is $310080 \mathrm{BTU}_{2}$ (from Ref. 6, page E2-1, assumed to be the same at OC), heat release rate 110 BTUs/sec-ft ${ }^{2}$ (from Ref. 2, Table 2E). The potential oil spread area around the pump was estimated at 5 $\mathrm{ft}^{2}$. The closest thermo-Lag wrap conduits run along the north wall about 20 ft . away longitudinally from the pump and at the highest elevation 4 ft . below the celling. Source (i.e., pump) was modeled as being 'in-center'.

## Results

Thermo-Lag target is not damaged (i.e., temperature does not reach $1000^{\circ} \mathrm{F}$ ) by HGL contributions, see Table 4. There just isn't enough energy in 2 gallons of oil. The maximum temperature at the T-L wrap was calculated to be $105^{\circ} \mathrm{F}$. A sample model worksheet is provided in Appendix A, Figure A-4.

Table 4. Temperature-Time History for Fire Scenario 4, Location No. 6.

| Time | HRR to | HRR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
| (min.) | TBrget | HGL |  |  |  |
| 0 | 550 | $($ Btu/s $)$ | $($ Btu $)$ | Target $-($ F $)$ | HGL-(F) |
| 6 | 550 | 550 | 0 | 100 | 100 |
| 10 | 550 | 550 | 198000 | 103 | 103 |
| 11 | 0 | 0 | 330000 | 105 | 105 |
| 20 | 0 | 0 | 330000 | 105 | 105 |
| 30 | 0 | 0 | 330000 | 105 | 105 |
| 36 | 0 | 0 | 330000 | 105 | 105 |
| 42 | 0 | 0 | 330000 | 105 | 105 |
| 48 | 0 | 0 | 330000 | 105 | 105 |
| 54 | 0 | 0 | 330000 | 105 | 105 |
| 60 | 0 | 0 | 330000 | 105 | 105 |
|  | 0 | 0 | 330000 | 105 | 105 |

Scenario \#1. The bounding scenario for this T-L configuration run was identified as the electrical panel V50F as the fixed ignition source during walkdowns. This panel is 6 feet off the floor and this was assumed as the virtual surface of the fire. The T-L condult runs horizontally along the wall 52 inches above this panel. The T-L wrap is approximately 3 inches in diameter, 0.5 inches thick, and was modeled "in-plume" of fire "against the wall".

## Results

As shown in Table 5, the energy contribution from the panel as modeled immediately raises the "in-plume" temperature to $1600^{\circ} \mathrm{F}$ at the target and is assumed to ignite the T-L wrap. The heat release rate assumed for this electrical cabinet is $400 \mathrm{BTUs} / \mathrm{sec}$ (containing non-qualified cable, Ref. 1, Appendix 1-B). A fire duration of 15 minutes was assumed for this electrical cabinet. It is assumed that 2 feet of T-L wrap with a surface area of $1.6 \mathrm{ft}^{2}$ ( 3 inches diameter) begins to buin. From NEI study on Thermo-Lag 330-1 Combustibility (Ref. 5 - Thermo-Lag 330-1 Combustibility, Attachment 1 and Table 1 of Attachment 3), heat release rates and total BTU for the T-L wrap were obtained ( $8.8 \mathrm{BTU} / \mathrm{sec}-\mathrm{ft}^{2}$ and $47,600 \mathrm{BTU} / \mathrm{in}$ aar foot, respectively). The T-L heat release rate of 14 BTU/sec was added to that of the electrical cabinet ( 400 BTU/sec) for HGL calculations. The virtual surface of the fire is at electrical cabinet 6 feet off the floor.

The cabinet fire is exhausted at time $t=15$ minutes. At this time, the $T$ - temperature (in plume) is $1600^{\circ} \mathrm{F}$. With the T-L wrap burning, the HGL temperature is only $108^{\circ} \mathrm{F}$. Ref. 1 , Appendix I-D indicates that "once Thermo-Lag is ignited, there is no test data to demonstrate how and when it will stop burning after the fire source is removed. Therefore, the Hazard tool assumes that once the fire source is removed, the Thermo-Lag in the plume (or ceiling jet) could be exposed to temperatures ranging from HGL (if the Thermo-Lag stops burning completely) to peak piume temperature (if the Thermo-Lag continues to burn)". It was very conservatively assumed here that the Thermo-Lag will continue to burn at peak plume temperature for the remainder of the scenario. A sample model worksheet is provided as Appendix A, Figure A-5.

Table 5. Temperature-Time History for Fire Scenario 1, Location No. 7.

| Time | HRR to | HRR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL |  |  |  |
| (min.) | $($ Btu/s) | (Btu/s) | $($ Btu) | Target-(F) | HGL-(F) |
| 0 | 400 | 400 | 0 | 1600 | 100 |
| 6 | 400 | 414 | 149040 | 1600 | 103 |
| 12 | 400 | 414 | 298080 | 1600 | 106 |
| 15 | 400 | 414 | 372600 | 1600 | 108 |
| 16 | 0 | 14 | 373440 | 1600 | 108 |
| 25 | 0 | 14 | 381000 | 1600 | 108 |
| 36 | 0 | 14 | 390240 | 1600 | 108 |
| 42 | 0 | 14 | 395280 | 1600 | 109 |
| 48 | 0 | 14 | 400320 | 1600 | 109 |
| 54 | 0 | 14 | 405360 | 1600 | 109 |
| 60 | 0 | 14 | 410400 | 1600 | 109 |

## Location No. 8

Scenario \#1. At this location, there is one vertical T-L condutt run that is by the equipment hatch. An open top cable tray (TGXR 2006) runs vertically right next to the T-L conduit 6 inches apart at its closest point. There are no other fixed ignition sources nearby except the RBCCW pump ( $\mathrm{P}-5-2$ ) which is more than 20 feet away.

Results

The analysis, showit in Figure A-R1, used FIVE Worksheet \#3 (Ref. 2) to model this scenario. It was assumed that 2 告 of 12 -inch wide cable tray (TGXR 2006) will self-ignite. A heat release rate of 23.4 BTU/sec- $\mathrm{ft}^{2}$ ( Ref. 1, FIVE Table 1E, cable $\# 5$ adjusted to consider Equation 2 on FIVE manual, page $10.4-10$ ) was assumed for the burning cable tray. The total HRR for this source is $47 \mathrm{BTU} / \mathrm{sec}$. The results indicate that if the cable tray is locatechat 0.8 fi or less of the T-L wrap, the T-L wrap reaches a critical damage flux of 2.2 BTU/sec-ft (this is the minimum ignition radiant flux for T-L, per Ref. 5 , page A1-8). This analysis is extremely conservative because it uses a point source radiant heat flux model (for the 2 feet of cable tray) and it assumes 40 percent of the total heat release rate of accidental fires is radiative, with the remainder released convectively.

## 5.

Fire modeling was performed for three locations containing Thermo-Lag fire barrier in this fire zone. Fire scenarios involving fixed and transient sources were analyzed. The analysis results are presented as temperature-time histories in Tables 1 through 5. These results are used in the following section to develop Total Heat Load and then convert to equivalent ASTM E-119 exposures.

## 6. APPROXIMATING EXPOSURE TIME-HISTORY OF THERMO-LAG SUBCONFIGURATIONS - THE TOTAL HEAT LOAD CONCEPT

Calculation of an exposure time-temperature profile for a specific T-L. Sub-configuration is based on the methodology presented in Reference 1. The time -temperature profiles are used to develop the total heat load (Reference 1, Appendix III-A) for each fire exposure. The total heat load concept holds that the area under the incident heat fiux vs. time curve, or total heat load, can be used as a measure of fire severity. Comparing the total heat load of a fire scenario exposure to the total heat load of a test exposure (ASTM E-119) allows one to predict the response of a Thermo-Lag barrier to the scenario exposure. The total heat load per unit area of fire barrier surface is equal to the area under the incident heat flux-time curve, corrected for convection effects as described in Reference 1, Appendix III-A.

### 6.1 Total Heat Load Results

| Location <br> No.(GA <br> DWG, <br> App. D) | Scenario | Description | Peak <br> Temperature <br> at Target <br> ( ${ }^{\circ}$ ) | Total Reat Load <br> Equivalent E. <br> 119 Exposure <br> Duration (min.) |
| :--- | :--- | :--- | :--- | :--- |
| 6 | 1 | cable Tray | 1593 | $\ll$ |
| 6 | 2 | Cable Tray | 1600 | $>60$ |
| 6 | 3 | Wooden Ladder | 1066 | $<18$ |
| 6 | 4 | CS Booster Pump | 105 | $<10$ |
| 7 | 1 | ElectricalPanel | 1600 | $>60$ |
| 8 | 1 | Cable tray | Radiant <br> Exposure | $>0.8 \mathrm{Ft}$ |

Appendix C contains tables with Total Heat Load values calculated for Thermo-Lag scenarios.

## 7. REFERENCES

1. EPRI Report "Methods for Evaluation of Cable Wrap Fire Barrier Performance", July, 1995
2. EPRI TR-100370, "Fire-Induced Vulnerability Evaluation," April 1992.
3. GPUN, "Fire Hazards Analysis Report - OC," Doc. No. 990-1746, Rev. 8, May 24, 1995.
4. GPUN - OC Procedure No. 120.5 (Revision 5), May 20, 1993.
5. Thermo-Lag 330-1 Combustibility Study, NEI, 1994.
6. GPUN - TMI-1 Procedure No. 1035 (Revision 24), August 23, 1994.
7. Personal communication with Maicom Gonzales of GPUN, in charge of plant oil program.

Appendix A

## Scenarios FIVE Worksheets

Figure A-1

| Temperature calculation at time |  | min | 6 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Ares |  | $f 2$ | 8100 |
| 1 | Location of Target |  | Plume |
| 2 | Height of Target above Fire Source | $f$ | 7.00 |
| 3 | Height from Fire Source to Ceiling, H | $f$ | 12.00 |
| 4 | Ratio of Target Height/Ceiling Height |  | 0.583 |
| 5 | Longitudinal Distance from Fire Source to Target, L | ft | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | f | NA |
| 8 | Height to Width Ratio, HWW |  | NA |
| 9 | Peak Fire Intensity | Btu/s | 141 |
| 10 | Fire Location Factor |  | 2 |
| 11 | Effective Fire intensity | Btu/s | 282 |
| 12 | Plume Temperature Rise at Target | F | 571 |
| 13 | Temp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | $F$ | 571 |
| 15 | Temperature at Target | $F$ | 672 |
| 16 | Temperature in HGL | F | 102 |
| 17 | Qtot to HGL | Btu | 50760 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated Qne! | Btu | 3046 |
| 20 | Caiculated Enclosure Volume, V | $f 13$ | 109200 |
| 21 | Calculated Qnei/V | Qnetit3 | 0.03 |
| 22 | HGL Temperature increase | F | 2 |
|  |  |  |  |

Figure A-2

| Temperature calculation at time |  | min | 6 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | 12 | 400 |
| 1 | Location of Targe! |  | Plume |
| 2 | Height of Target above Fire Source | f | 8.00 |
| 3 | Height from Fire Source to Celling, H | fif | 18.00 |
| 4 | Ratio of Target HeighvCeiling Height |  | 0.444 |
| 5 | Longitudinal Distance from Fire Source to Target, L | $\pi$ | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | f | NA |
| 8 | Height to Width Ratio, H/W |  | NA |
| 9 | Peak Fire Intensity | Btw/s | 328 |
| 10 | Fire Location Factor |  | 2 |
| 11 | Effective Fire Intensity | Btu/s | 656 |
| 12 | Plume Temperature Rise at Target | F | 802 |
| 13 | Temp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | $F$ | 802 |
| 15 | Temperature at Target | $F$ | 963 |
| 16 | Temperature in HGL | F | 161 |
| 17 | Qtot to HGL | Btu | 118080 |
| 18 | Estirnated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated Qnet | Btu | 7085 |
| 20 | Calculated Enclosure Volume, V | A3 | 7200 |
| 21 | Calculated Qnet/V | Qnei/f 3 | 0.98 |
| 22 | HGL Temperature Increase | F | 61 |
|  |  |  |  |

Figure A-3

| Temperature calculation at time |  | min | 6 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | 12 | 400 |
| 1 | Location of Target |  | Plume |
| 2 | Height of Target above Fire Source | $f$ | 4.00 |
| 3 | Height from Fire Source to Ceiling, H | ก | 24.00 |
| 4 | Ratio of Target Height/Ceiling Height |  | 0.167 |
| 5 | Longitudinal Distance from Fire Source to Target, L | fi | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | f | NA |
| 8 | Height to Width Ratio, H/W |  | NA |
| 9 | Peak Fire Intensity | Btw/s | 214 |
| 10 | Fire Location Factor |  | 1 |
| 11 | Effective Fire Intensity | Btw/s | 214 |
| 12 | Plume Temperature Rise at Target | F | 1207 |
| 13 | Temp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | F | 1207 |
| 15 | Temperature at Target | F | 1336 |
| 16 | Temperature in HGL | $F$ | 129 |
| 17 | Qtot to HGL | Btu | 77040 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated Qnet | Btu | 4622 |
| 20 | Calculated Enclosure Volume, V | $f 13$ | 9600 |
| 21 | Calculated Qnet/V | Qnet/fi3 | 0.48 |
| 22 | HGL Temperature increase | F | 29 |
|  |  |  |  |

Figure A-4

| Femperature catculation at time |  | min | 6 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | $f 12$ | 9100 |
| 1 | Location of Target |  | Hot Gas Layer |
| 2 | Height of Target above Fire Source | fi | 20.00 |
| 3 | Height from Fire Source to Ceiling, H | fi | 24.00 |
| 4 | Ratio of Target Height/Celling Height |  | 0.833 |
| 5 | Longitudinal Distance from Fire Source to Target, L | $f$ | NA |
| 6 | Longitudinal Distance to Height Ratio, 1 / |  | NA |
| 7 | Enclosure Width, W | f | NA |
| 8 | Height to Width Ratio. H/W |  | NA |
| 3 | Peak Fire Intensity | Btu/s | 550 |
| 10 | Fire Location Factor |  | 1 |
| 11 | Effective Fire Intensity | Btu/s | 550 |
| 12 | Plume Temperature Rise at Target | $F$ | 155 |
| 13 | Temp Rise Factor at Target |  | 0.00 |
| 14 | Temperature Rise at Target | $F$ | 0 |
| 15 | Temperature at Target | $F$ | 103 |
| 16 | Temperature in HGL | $F$ | 103 |
| 17 | Qtot to HGL | Btu | 198000 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated Qnet | Btu | 11880 |
| 20 | Calculated Enciosure Volume, V | $f 13$ | 218400 |
| 21 | Caiculated Qnet/V | Qnet/f3 | 0.05 |
| 22 | HGL. Temperature increase | $F$ | 3 |
|  |  |  |  |

Figure A-5

| Temperature calculation at time |  | min | 6 |
| :---: | :---: | :---: | :---: |
| Maximum am'sent temperature |  | F | 100 |
| Floor Area |  | $f 12$ | 8100 |
|  |  |  |  |
| 1 | Location of Target |  | Plume |
| 2 | Fisight of Target above Fire Source | fi | 4.33 |
| 3 | Fheight from Fire Source to Celiling, H | f | 18.00 |
| 4 | Ratio of Target Height/Ceiling Height |  | 0.241 |
| 5 | Longitudinal Distance from Fire Source to Target, L | $f$ | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | $f$ | NA |
| 8 | Height to Widith Ratio, HWW |  | NA |
| 9 | Peak Fire Intensity | Btw/s | 400 |
| 10 | Fire Location Factor |  | 2 |
| 11 | Effective Fire Intensity | Btu/s | 800 |
| 12 | Plume Temperature Rise at Target | F | 1600 |
| 13 | Temp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | $F$ | 1600 |
| 15 | Temperature at Target | F | 1600 |
| 16 | Temperature in HGL | F | 103 |
| 17 | Qtot to HGL | Btu | 149040 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 18 | Calculateci Qnet | Btu | 8942 |
| 20 | Calculated Enclosure Volume, V | $f 13$ | 163800 |
| 21 | Calculated QneiV | Qnet/f3 | 0.05 |
| 22 | HGL Temperature Increase | F | 3 |
|  |  |  |  |

Figure A-R1, scenario 1, Loc. No. 8

## WORKSHEET 3: RADIANT EXPOSURE SCENARIOS

| 1 | Critical Radiant Flux to the Target <br> (Representative value $=1$ ) | Btw/s/R2 | 2.2 |
| :--- | :--- | :---: | :---: |
| 2 | Peak Fire Intensity | Btw/s | 47 |
| 3 | Radiant Fraction of Heat Release <br> (Representative value $=0.4)$ | -User Input |  |
| <-User Input |  |  |  |
| c-User Input |  |  |  |

If the expowure fire is lockted whinin this distance (indicated in Bou 5) of the target, erticel condilions can ocour. Outaide this range, cribical conditione are not indicated for the wownario under consuidenation.

## Appendix B

Basis for Following ASTM E-119 Time-Temperature Relationship When T(HGL) is $1000^{\circ} \mathrm{F}$

## TRANSITION TO

ASTM E-119 TIME-HISTORY
IN THE HOT GAS LAYER

## ISSUE:

The Fire Hazard Tool currently under development in the EPRI Cable Wrap Fire Barrier Tailored Collaboration assumes that the fire compartment temperature will follow the ASTM E119 time-temperature relationship as soon as the compartment hot gas layer reaches $1000^{\circ} \mathrm{F}$.

BASIS:

- The Handbook (Ref, pg 6-75) states that *Flashover of an enclosure is likely to occur HI the temperature of the upper gas layer reaches approximately $1000^{\circ} \mathrm{F}\left(600^{\circ} \mathrm{C}\right)$.

The EPRI assumption of $1000^{\circ} \mathrm{F}$ is therefore conservative.

- As stated near the top of page 6-67 of the Handbook, the intensity of the fire will be somewhat lower when the walls and ceiling absorb signifficant amounts of energy rather than when act as insulation or radiation barriers.

Since power plant compartments are typically concrete barriers which absorb significant amounts of heat, the intensity of fire will tend to be less severe than fires in compartments with insulated barriers.

- ASTM E-119 has been the accepted standard for testing fire barrier systems since the early 1900's. The standard, which was developed from actual fire tests, has been used to evaluate the fire endurance capabilities of fire barriers since it was developed. The time-temperature relationship represented by the curve represents a fully-involved fire compartment.
- According to Fred Mowrer of the University of Maryland, plume and ceiling effects disappear in a fully involved fire compartment as the plume/ceiling jet gas are no longer more buoyant than surrounding gases.
- Use of the ASTM E-119 time-temperature relationship to represent a fully involved power plant compartment fire is therefore consistent with industry practice. Assuming that that occurs when the hot gas layer reaches $1000^{\circ} \mathrm{F}$ is more conservative than the $1100^{\circ} \mathrm{F}$ noted in the Handbook.

Reference: NFPA Fire Protection Handbook, Seventeenth Edition, Section 6/Chapter 6.

## Appendix C

Total Heat Load Results

Figure C -1

Total Heat Load
Scenario 1, Loc. No. 6


Figure C-2

Total Heat Load
Scenario 2, Loc. No. 6

|  |  |  | Target | Hot Gas Layer | ASTM E-118 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btu/s-si) | $(1000$ Btu/s? | $(1000$ Btu/s? | $(1000$ tw/sf) |
| 0 | 902 | 2.1 | 0.0 | 0.00 | 0.00 |
| 6 | 963 | 2.5 | 0.83 | 0.13 | 0.49 |
| 12 | 1212 | 4.3 | 2.06 | 0.27 | 1.35 |
| 18 | 1493 | 7.8 | 4.24 | 0.44 | 3.56 |
| 24 | 1600 | 9.5 | 7.34 | 0.72 | 5.97 |
| 30 | 1600 | 9.5 | 10.75 | 1.29 | 8.84 |
| 36 | 1600 | 9.5 | 14.16 | 2.67 | 11.59 |
| 42 | 1600 | 9.5 | 17.57 | 4.84 | 14.73 |
| 48 | 1600 | 8.5 | 20.97 | 7.31 | 18.03 |
| 54 | 1600 | 8.5 | 24.38 | 10.10 | 21.49 |
| 60 | 1600 | 8.5 | 27.79 | 13.12 | 25.15 |

Figure C-3

Total Heat Load
Scenario 3, Loc. No. 6

|  |  |  | Target | Hot Gas Layer | ASTM E-118 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btu/s-sf) | (1000Btu/s') | (1000Btu/st) | (1000Btu/st) |
| 0 | 100 | 0.4 | 0.0 | 0.00 | 0.00 |
| 6 | 129 | 0.3 | 0.12 | 0.12 | 0.41 |
| 12 | 166 | 0.4 | 0.23 | 0.23 | 1.65 |
| 18 | 213 | 0.3 | 0.35 | 0.35 | 3.56 |
| 24 | 272 | 0.5 | 0.48 | 0.48 | 5.83 |
| 30 | 345 | 0.5 | 0.88 | 0.66 | 8.64 |
| 36 | 435 | 0.6 | 0.87 | 0.87 | 11.58 |
| 42 | 545 | 0.0 | 1.14 | 1.14 | 14.73 |
| 48 | 683 | 1.3 | 1.52 | 1.52 | 18.03 |
| 54 | 853 | 1.9 | 2.10 | 2.10 | 21.48 |
| 60 | 1066 | 3.2 | 3.02 | 3.02 | 25.15 |

Figure C-4

Total Heat Load
Scenario
4, Loc. No. 6

|  |  |  | Target | Hot Gas Layer | ASTM E-119 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btu/s-st) | (1000Btw/sf) | (10008tu/s? | (1000Btu/s) |
| 0 | 100 | 0.4 | 0.0 | 0.00 | 0.00 |
| 6 | 103 | 0.2 | 0.11 | 0.11 | 0.41 |
| 10 | 105 | 0.2 | 0.16 | 0.16 | 1.24 |
| 11 | 105 | 0.2 | 0.18 | 0.18 | 1.52 |
| 20 | 105 | 0.2 | 0.30 | 0.30 | 4.57 |
| 30 | 105 | 0.2 | 0.44 | 0.44 | 8.83 |
| 36 | 105 | 0.2 | 0.53 | 0.53 | 11.88 |
| 42 | 105 | 0.2 | 0.61 | 0.64 | 15.02 |
| 48 | 105 | 0.2 | 0.70 | 0.70 | 18.32 |
| 54 | 105 | 0.2 | 0.78 | 0.78 | 21.77 |
| 60 | 105 | 0.2 | 0.87 | 0.87 | 25.44 |

Figure C-5

Total Heat Load

|  |  |  | Target | Hot Gas Layer | ASTM E-118 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btus-sf) | (1000Btu/sf) | (1000Btu/s) | (1000Btu/st) |
| 0 | 1600 | 9.5 | 0.0 | 0.00 | 0.00 |
| 6 | 1600 | 9.5 | 3.41 | 0.11 | 0.41 |
| 12 | 1600 | 0.5 | 6.82 | 0.18 | 1.65 |
| 15 | 1600 | 8.5 | 8.52 | 0.23 | 2.59 |
| 16 | 1600 | 9.5 | 9.08 | 0.25 | 2.94 |
| 25 | 1600 | 9.5 | 14.20 | 0.38 | 6.53 |
| 36 | 1600 | 9.5 | 20.46 | 0.54 | 11.73 |
| 42 | 1600 | 9.5 | 23.86 | 0.62 | 14.87 |
| 48 | 1600 | 8.5 | 27.27 | 0.71 | 18.17 |
| 54 | 1600 | 9.5 | 30.68 | 0.80 | 21.63 |
| 60 | 1600 | 9.5 | 34.09 | 0.88 | 25.29 |

Appendix D

## General Arrangement Drawings



## FIRE ZONE

## RB-FZ-1D



## TR 102 <br> APPENDIX B

# Using FIVE Fire Modeling to Approximate Exposure TimeHistory for Thermo-Lag Configurations in OC Fire Zone RB-FZ-1E 

Letter Report<br>September 27, 1995

Prepared by:
GPUN Risk Analysis Group
Using FIVE Fire Modeling to Approximate Exposure Time-History for Thermo-Lag Configurations

## 1. PURPOSE

The purpose of this report is to document the detalled fire modeling performed for fire scenarios involving Thermo-Lag wrapped fire barriers, as identified during plant walkdowns of OC fire zone RB-FZ-1E. Thermo-Lag exposure time-history results were obtained for each T-L configuration determined to be impacted by a fire source/combustible.

## 2. APPROACH

The approach followed for modeling of fire scenarios is based on methodology detailed in the EPRI "Methods for Evaluation of Cable Wrap Fire Barrier Performance* document (Ref. 1) which was based on fire modeling techniques developed for FNE (Ref. 2). Fire modeling was accomplished in two steps:

1. Screening walkdowns to eliminate from further consideration those fire ignition sources and potential fire scenarios that cannot develop into damaging fires.
2. Fire modeling of specific scenarios that were not screened out in the first step.

For each Thermo-Lag configuration analyzed, one or more fire scenarios (including bounding scenarios) were postulated based on the following:
a. information gathered during plant walkdowns (L.e., fixed or transient ignition source and Thermo-Lag target location, type and quantity of combustibles present in zone);
b. information obtained from reviewing applicable sections in Appendix R documents for OC (Ref. 3);
c. information obtained from reviewing OC procedure 120.5, Rev. 5, "Control of Combustibles." (Ref. 4)
d. information provided by GPUN engineers (specialling in fire protection, risk assessment, mechanical).

Thermo-Lag FIVE analyses were performed using computerized Excel spreadsheets which duplicate FIVE Worksheets 1,2 , or 3 and are linked to tables containing the same information as in FIVE Reference Tables. For each Thermo-Lag fire modeling scenario, results are presented as exposure time-temperature profiles. In all scenarios evaluated, time-dependent temperatures were derived by incrementally adding the total heat content of the fuel into the fire compartment hot gas layer.

## 3. FIRE MODELING ASSUMPTIONS

The following Thermo-Lag fire modeling assumptions were made:

1. Worst-cass fixed fire source scenarios were modeled for each Thermo-Lag subconfiguration. The determination of which fixed fire source might cause the worstcase Thermo-Lag fire scenario was made during the plant fire area walkdown and
was based on minimum damage threshold heights and distances calculated for typical fixed fire sources (e.g., electrical cabinets, electrical motors, lube oil in pumps) found at Oyster Creek.
2. Worst-case transient fire source scenarios observed during walkdowns were also modeled. Worst-case transient fire scenarios considered were similar to the type and quantities of transient combustibles allowed by OC station procedure (Ref. 4).
3. To ensure conservative results, worst-case fire plume (or ceiling jet) scenarios and/or radiant flux scenario were first modeled. For TL sub-configurations not in the plume or ceiling jet, hot gas layer fire scenarios were evaluated.
4. Electrical cable inside cabinets, motor control centers, switchgear and cable trays was assumed to be non-qualified (i.e., non IEEE-383 type). This assumption is conservative since electrical cable purchased and instalied at Oyster Creek in the past decade was IEEE-383 type.
5. The Fire Hazard Tool developed for the EPRI Cable Wrap Fire Barrier Tailored Collaboration assumes that the fire compartment temperature will follow the ASTM E-119 time-temperature curve as soon as the compartment hot gas layer (HGL) reaches $1000^{\circ} \mathrm{F}$. When the HGL temperature reaches $1000^{\circ} \mathrm{F}$, the plume (or ceiling jet) temperatures are ascumed to follow that of the E-119 (see discussion in Appendix B).
6. The Thermo-Lag wrap is assumed to burn by itself (i.e., even after the initial fire source is out of fuel) unless otherwise stated. Per, the EPRI Method (Ref. 1, Appendix I-D), credit may be taken for a time lag before the Thermo-Lag ignites based on exposure temperature.
7. The maximum plume and ceiling jet temperature is assumed to be $1600^{\circ} \mathrm{F}$, which corresponds to flame temperature. This assumption is valid until hot gas layer temperature exceeds $1600^{\circ} \mathrm{F}$, at which point the plume temperature is assumed to be equal to hot gas layer temperature.
8. Fire Zone RB-FZ-1E specific data:

- total area of 12,140 th2 (Ref. 3 - Oyster Creek Fire Hazard Analysis Report, Rev. No. 8);
- fire zone ceiling height of 25 feet (Oyster Creek General arrangement Drawings);
- ambient temperature of $100^{\circ} \mathrm{F}$.


## 4. FIRE MODELING RESULTS

### 4.1 Fire Zone RB-FZ-1E

Fire zone RB-FZ-1E is the Reactor Building 23' Elevation. Fire modeling was performed for three areas on Elevation $23^{\prime}-6^{\prime \prime}$ and two areas on Elevation $33^{\prime}-5^{\prime \prime}$ in this fire zone that contain the T-L wrap. They are designated as location Nos. 10, 11, 12, 9, and 15 on the general arrangement drawings shown in Appendix D .

## Location No. 10

Two scenarios were modeled for this T-L configuration. There are two vertical runs of T-L
wrapped condults about 4 ft . long at elevation 12 ft . above the fioor at this location. One condutt is $11 / 2^{\prime \prime}$ with $1 / 2^{\prime \prime}$ thick T-L wrap and the other is $3^{\prime \prime}$ with $1 / 2^{\prime \prime}$ thick T-L wrap. The first scenario models a transient ignition source/combustible and the other evaluates the impact of scenario \#1 for location \#11 on the T-L configuration here.

Scenario \#1. This fire scenario modeled a transient ignition source/combustible, a gallon container containing lube oll. Per Oyster Creek Procedure "Control of Combustibles," Procedure No. 120.5, Rev. 4 (Ref. 4), combustible liquids could be transported throughout the plant in an approved closed metal container (but not necessarily a safety-can). For this fire scenario, It was postulated that 1 quart of lube oll is spilied out of a can on the fioor and is not cleaned up. The oil is later ignited by a welder working nearby or another transient ignition source and begins to burn. The oil fire could potentially impact a T-L wrap 12. feet above.

The heat release rate for the lube oil is $110 \mathrm{Btu} / \mathrm{s}-\mathrm{ft}^{2}$ (from Ref. 2, Table 2E). The oil could spread unconfined over an area of $13.5 \mathrm{ft}^{2}$ ( $54 \mathrm{ft}^{2} / \mathrm{gal}$ trom Ref. 2, Table 3E). The heat release rate assumed for this fire scenario is $1,485 \mathrm{Btu} / \mathrm{s}\left(110 \mathrm{Btu} / \mathrm{s}-\mathrm{ft}^{2} \times 13.5 \mathrm{ft}^{2}\right.$ ). The heat content of one quart of oil is ( $17,111 \mathrm{Btu} / \mathrm{bb}$. ${ }^{*} 60 \mathrm{lb} . / 1 \mathrm{ft}^{3} .{ }^{*} 1 \mathrm{ft}^{3} / 30$ quarts) $34,222 \mathrm{Btu}$ (from Ref. 2, Table 2E). The oil fire duration will be less than one minute, but one minute will be used for conservatism. The virtual surface of the fire is at the floor level. The location factor for this source is one.

## Results

The short oil fire does not cause an exposure "in-plume" temperature high enough to ignite the Thermo-Lag. The fire is of a short duration, with very slight hot gas layer effects. A sample model worksheet is provided as Appendix A, Figure A-1.

Table 1. Temperature-Time History for Fire Scenario 1, Location No. 10.

| Tirne | HRR io | HRR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL |  |  |  |
| (min.) | $($ Btu/s $)$ | $($ Btu/s $)$ | $($ Btu) | Target - $(\mathbf{F})$ | HGL - $(\boldsymbol{F})$ |
| 0 | 1485 | 1485 | 0 | 804 | 100 |
| 1 | 1485 | 1485 | 89100 | 805 | 101 |
| 2 | 0 | 0 | 89100 | 101 | 101 |
| 10 | 0 | 0 | 89100 | 101 | 101 |
| 15 | 0 | 0 | 89100 | 101 | 101 |
| 25 | 0 | 0 | 89100 | 101 | 101 |
| 30 | 0 | 0 | 89100 | 101 | 101 |
| 35 | 0 | 0 | 89100 | 101 | 101 |
| 40 | 0 | 0 | 89100 | 101 | 101 |
| 50 | 0 | 0 | 89100 | 101 | 101 |
| 60 | 0 | 0 | 89100 | 101 | 101 |

Scenario \#2. This scenario evaluates the impact of HGL produced by the self ignition of stack of four cable trays described in scenario \#1 for location \#11 on the T-L wrap in this location \# 10.

## Results

This fire zone is pretty open and location nos. 10 and 11 freely communicate with each other, therefore, the HGL temperatures calculated for scenario \#1 for location \#11 is also assumed to apply here. It should be noted that the bottom cable tray, the auto-ignition sourca/combustible for that scenario, is at 12 ft . below the celling and the T-L. wrap here is at about 9 ft . below the ceiling (top of the T-L. wrapped condults), therefore, the target is in the HGL region of scenario \#1 for location \#11. The results are presented in Tabie 2.

Table 2. Temperature-Time History for Fire Scenario 2, Location No. 10.

| Time | HRR to | HRR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL |  |  |  |
| (min.) | $($ Btu/s $)$ | $($ Btu/s $)$ | $(B t u)$ | Target-(F) | HGL-(F) |
| 0 | 1047 | 1047 | 0 | 100 | 100 |
| 6 | 1234 | 1234 | 444240 | 111 | 111 |
| 12 | 1421 | 1421 | 955800 | 124 | 124 |
| 18 | 1608 | 1608 | 1534680 | 138 | 138 |
| 24 | 1795 | 1795 | 2180880 | 155 | 155 |
| 30 | 1982 | 1982 | 2894400 | 174 | 174 |
| 36 | 2169 | 2169 | 3675240 | 196 | 196 |
| 42 | 2356 | 2356 | 4523400 | 221 | 221 |
| 48 | 2543 | 2543 | 5438880 | 248 | 248 |
| 54 | 2730 | 2730 | 6421680 | 279 | 279 |
| 60 | 2917 | 2917 | 7471800 | 313 | 313 |

## Location No. 11

Three scenarios were mcdeled for this T-L configuration. Three T-L wrapped $1^{*}$-condults in this area run along the ceiling (CGCR3021, CGFR3019, \& CNXR2050, see B\&R Drawing Nos. $1303 \& 1317$ ). There are four $24^{4}$-wide cable trays stacked on top of each other that run directly underneath this T-L configuration. In the north view, there is a $24^{4}$-wide cable tray that runs about 4 -inches apart from the T-L wrapped penetration box on the drywell at about 8 ft . below the ceiling. This was found to be the most limiting case from all the four T-L wrapped penetration boxes in this area (Penetration Nos. 8, 9, 18, \& 19). There is also a core spray booster pump on the floor at this elevation that is 15 ft . away longitudinally from the T-L wrapped condults on the ceiling.

Scenario \#1. The fixed ignition source considered here, is the self-ignition of stack of four cable trays (Nos. 23, 25, 24, \& 26). They are stacked one on top of each other about 1 ft . apart with the closest one (cable tray \#23) being about 9 ft . directly below the T-L wrap conduit runs on the ceiling. All the cable trays are 24 inches wide and cable trays \#24 and \# 26 (the bottom two trays) have light loads.

This scenario models sell-ignition of 2 ft . of cable tray \# 26 initially which will then ignite 4 ft . of cable tray \# 24, 6 ft. of cable tray \#25, and 8 ft of cable tray \# 23. Ht is conservatively assumed that all the cable trays ignition occur instantaneously for calculation simplification. In addition to the vertical propagation, the fire is also assumed to propagate horizontally along the exposed cable tray with a flame speed of 10 feet per hour (per Ref. 1, Appendix I-C). The total amount of cable tray available for this fire scenario is 10 feet per tray (for the first hour of fire scenario). The T-L target was assumed "in-plume" with the fire ignition source modeled as "in-center".

The heat release rates considered for this scenario before the T-L wrap is ignited are as follow (by fire source):

1. first cable tray (\#26) - 23.4 BTU/sec- fi2 (Ref. 2, FIVE Table 1E, cable \#5 adjusted to consider Equation 2 on FNE manual, page 10.4-10); since 4 th2 are initially burning, the HRR for this source alone is 93.6 BTU/sec; this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be $140.4 \mathrm{BTU} / \mathrm{s}$, which includes the HRR from 2 t 2 of additional cable tray);
2. second cable tray (\# 24) - 23.4 BTU/sec- ft2 (Ref. 2, FNE Table 1E, cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); just as with the first cable tray, the HRR for this source alone is 187 BTU/sec (since 8 ft2 are initially burning); this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be 234 BTU/s, which includes the HRR from 2 tt2 of additional cable tray);
3. third cable tray (\#25) - 23.4 BTU/sec- fi2 (Ref. 2, FIVE Table 1E, cable \#5 adjusted to consijer Equation 2 on FIVE manual, page 10.4-10); just as with the second cable tray, the HRR for this source alone is 281 BTU/sec (since 12 ft2 are initially burning); this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be 328 BTU/s, which includes the HRR from $2 \mathrm{ft2}$ of additional cable tray);
4. forth cable tray (\#23) - 23.4 BTU/sec-ft2 (Ref. 2. FIVE Table 1E, cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); just as with the second cable tray, the HRR for this source alone is 374 BTU/sec (since $16 \mathrm{ft2}$ are initially burning); this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be $421 \mathrm{BTU} / \mathrm{s}$, which includes the HRR from 2 ft 2 of additional cable tray);

The total available heat (Otot) from each fire source is:

1. first cable tray (\#26) $-2,000,000$ BTUs [it assumes $10,000 \mathrm{BTU} / \mathrm{bm}$ of cable insulation (per Ref. 6, page E2-1) $\times 10$ linear feet $\times 20 \mathrm{lbm} / \mathrm{linear}$ foot (engineering judgment for $50 \%$ filled, 24 inch wide cable tray)];
2. second cable tray (\#24) $-2,000,000$ BTUs [it assumes 10,000 BTU/lbm of cable insulation (per Ref. 6, page E2-1) $\times 10$ linear feet $\times 20 \mathrm{lbm} / \mathrm{linear}$ foot (engineering judgment for $50 \%$ filled, 24 inch wide cable tray)];
3. third cable tray (25) $-4,000,000$ BTUs [itassumes 10,000 BTU/lbm of cable insulation (per Ref. 6, page E2-1) $\times 10$ linear feet $\times 40 \mathrm{lbm} / l i n e a r ~ f o o t ~(e n g i n e e r i n g ~ j u d g m e n t ~ f o r ~ 100 \% ~$ filled, $\quad 24$ inch wide cable tray)];
4. forth cable tray (\#23) $-4,000,000$ BTUs [Ht assumes 10,000 BTU/lbm of cable insulation (per Ref. 6, page E2-1) $\times 10$ linear feet $\times 40 \mathrm{lbm} / \mathrm{linear}$ foot (engineoriny judgment for $100 \%$ filled, 24 inch wide cable tray)];

Once the cable-trays' fire ignites the T-L wrapped conduits, they will start to burn. T-L wrap HRR is assumed to be 8.8 BTU/sec-tt2 (per Ref. 5); the amount of Thermo-Lag that could be initially involved in this fire scenario is assumed to be 24 linear feet ( 8 linear feet per conduit run that is directly above cable tray \#23). The total surface area is 12.6 ft (conduits are 1 . inch in diameter with $1 / 2$ inch thick T-L wrap); the HRR is thus $111 \mathrm{BTU} / \mathrm{sec}$. The impact of horizontal propagation of cable tray fire on the T-L wrap condults is ignored here due to insignificant heat contribution from the T-L wrap as compared to the cable trays. The total avallable heat (Qtotal) from this amount of T-L fire source is 571,200 BTUs (it assumes 23,800 BTU/linear foot, from Ref. 5).

## Results

The results are presented in Table 3 below. As ir cated by the temperature-time histories in the table, the 'in-plume' temperature at the T-L targets 9 feet above the burning cable trays reaches $1000^{\circ} \mathrm{F}$ immediately, so the T-L wrap will be ignited and burn. However, the HGL temperature is only $313^{\circ} \mathrm{F}$ after one hour into this scenario. A sample model worksheet is provided in Appendix A, Figure A-2.

Table 3. Temperature-Time History for Fire Scenario 1, Location No. 11.

| Time | HRR to | HRR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL. |  |  |  |
| (min.) | (BTU/s) | (Btu/s) | (BTU) | Target - (F) | HGL-(F) |
| 0 | 936 | 1047 | 0 | 1043 | 100 |
| 6 | 1123 | 1234 | 444240 | 1054 | 117 |
| 12 | 1310 | 1421 | 955800 | 1169 | 124 |
| 18 | 1497 | 1608 | 1534680 | 1281 | 138 |
| 24 | 1684 | 1795 | 2180880 | 1391 | 155 |
| 30 | 1871 | 1982 | 2894400 | 1500 | 174 |
| 36 | 2058 | 2169 | 3675240 | 1600 | 196 |
| 42 | 2245 | 2356 | 4523400 | 1600 | 221 |
| 48 | 2432 | 2543 | 5438880 | 1600 | 248 |
| 54 | 2619 | 2730 | 6421680 | 1600 | 279 |
| 60 | 2806 | 2917 | 7471800 | 1600 | 313 |

Scenario \#2. This scenario evaluates the radiant exposure from self-ignition of the $24^{*}$-wide cable tray running about 4 inches apart from the T-L wrapped penetration box on the drywell.

## Results

The analysis, shown in Appendix A, Figure A-R1, uses FIVE Worksheet \#3 (Ref. 2). It was assumed that 2 ft of 24 -inch wide cable tray will self-ignite. A heat release rate of 23.4 BTU/sec- $\mathrm{ft}^{2}$ (per Ref. 2, FIVE Table 1E, cable \#5 adjusted to consider Equation 2 on FIVE manual, page $10.4-10$ ) was assumed for the burning cable tray. The total HRR for this source is $93.6 \mathrm{BTU} / \mathrm{sec}$. The results indicate that if the cable tray is locatech at 1.2 ft or less of the T-L wrap, the T-L wrap reaches a critical damage flux of $2.2 \mathrm{BTU} / \mathrm{sec}-\mathrm{ft}^{2}$ (this is the minimum ignition radiant flux for T-L, per Ref. 5, page A1-8). This analysis is extremely conservative because it uses a point source radiant heat flux model (for the 2 feet of cable tray) and it assumes 40 percent of the total heat release rate of accidental fires is radiative, with the remainder released convectively.

Scenario \#3. The fixed ignition source modeled inere is the core spray bouster pump P-202B. This pump contains up to 2 gallons of lube oil (Ref. 7). Total Q available is $310080 \mathrm{BT} \mathrm{Z}_{2}$ (from Ref. 6, page E2-1, assumed to be the same at OC), heat release rate $110 \mathrm{BTUs} / \mathrm{sec}-\mathrm{ft}^{2}{ }_{2}$ (from Ref. 2, Table 2E). The potential oil spread area around the pump was estimated at $4 \mathrm{ft}^{2}$ during the plant walkdown. The Thermo-Lag wrap conduits are located on the ceiling 15 feet away longitudinally from the pump and in the "Ceiling Jet" sublayer. Source (i.e., pump) was modeled as being 'in-center'. Ambient temperature assumed was $100^{\circ} \mathrm{F}$.

Results

Thermo-Lag target is not damaged (i.e., temperature does not reach $1000^{\circ} \mathrm{F}$ ) by Celling Jet effects or Ceiling Jet/HGL. contribution. There just isn't enough energy in 2 galluns of oll. The maximum temperature at the T-L wrap was calculated to be $135^{\circ} \mathrm{F}$. Temperstire-time histories are shown in Table 4. A sample model worksheet is provided in Appendix A, Figure A-3.

Table 4. Temperature-Time Hietory for Fire Scenario 3, Location No. 11.

| Time | HRR to | HRR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL |  |  |  |
| (min.) | (Biu/s) | (Btu/s) | (Btu) | Target-(F) | HGL-(F) |
| 0 | 440 | 440 | 0 | 131 | 100 |
| 5 | 440 | 440 | 132000 | 132 | 102 |
| 10 | 440 | 440 | 264000 | 134 | 103 |
| 12 | 440 | 440 | 316800 | 135 | 104 |
| 13 | 0 | 0 | 316800 | 104 | 104 |
| 15 | 0 | 0 | 316800 | 104 | 104 |
| 20 | 0 | 0 | 318800 | 104 | 104 |
| 30 | 0 | 0 | 316800 | 104 | 104 |
| 40 | 0 | 0 | 316800 | 104 | 104 |
| 50 | 0 | 0 | 316800 | 104 | 104 |
| 60 | 0 | 0 | 316800 | 104 | 104 |

## Location No. 12

There is a 1 -inch T-L wrapped conduit (CNXR2051) and associated T-L wrapped penetration box (penetration 57) in this area (see B\&R drawing Nos. E1303 \& E1317). About five feet of a half full $24^{\prime \prime}$-cable tray runs directly underneath the penetration box, vertically on the drywell, placing the box right in the plume of such cable fire. Auto-ignition of this cable was modeled as one scenario for this area. Also, this area is very open and freely communicates with other areas in this fire zone, therefore, the HGL temperatures calculated for scenario \#1 for location \#11 is also applicable here. Finally, a transient ignition source/combustible was considered for this location to be placed on the floor against the drywell underneath the T-L wrapped conduit.

Scenario \#1. This scenario models auto-ignition of 5 ft . of a haff full $24^{\prime \prime}$-wide cable tray against the wall directly underneath the T-L wrapped penetration box. The penetration box was estimated to be 17 ft . above the floor. A heat release rate of $23.4 \mathrm{BTU} / \mathrm{sec}-\mathrm{ft}{ }^{2}$ (Ref. 2, FIVE Table 1E, cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); was used for this cabie tray. Since $5 \pi^{2}$ are initially assumed to be burning for the "in-plume" caiculations, the HRR for this source is $117 \mathrm{BTU} / \mathrm{sec}$. at the beginning of the scenario. The total available heat from this source is $4,000,000$ BTUs [tt assumes 10,000 BTU/bm of cable tray (per Ref. 6, page E2-1) $\times 20$ linear feet $\times 20 \mathrm{lbm} / \mathrm{linear}$ foot (engineering judgment for $50 \%$ filled, 24 inch wide cable tray)]. The target was assumed "in-plume" for this scenario.

## Results

The analysis as shown in Table 5 below indicate that the "in-plume energy contribution is able to immediately ignite the T-L. wrap and the temperatures reach $1600^{\circ} \mathrm{F}$ at the target. Once the cable-tray's fire ignites the T-L wrapped penetration box, It will start to burn. T-L wrap HRR is assumed to be 8.8 BTU/sec-ft2 with a heat load of 36,750 BTU//t2 for $1 / 2^{\circ}$ thick T-L wrap (per Ref. 5). The total surface area of the T-L wrap that would be in the plume of such fire is approximated at $2.5 \mathrm{ft2}$. The HRR is thus 22 BTU/sec., and the T-L. wrap should last for 70 minutes ( for a total heat load of 91875 BTUs). The "HGL" temperatures were calculated for this scenario considering that the HRR for the burning cable tray increases with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be $140.4 \mathrm{BTU} / \mathrm{s}$, which includes the HRR from 1 tit of additional cable tray). A sample model worksheet is provided as Appendix A, Figure A-4.

Table 5. Temperature-Time History for Fire Scenario 1, Location No. 12.

| Time | HRR to | HRR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL |  |  |  |
| (min.) | (BIu/s) | (Btu/s) | (Btu) | Target-(F) | HGL-(F) |
| 0 | 117 | 139 | 0 | 1600 | 100 |
| 6 | 117 | 162 | 58320 | 1600 | 102 |
| 12 | 117 | 185 | 125280 | 1600 | 105 |
| 18 | 117 | 209 | 200520 | 1600 | 107 |
| 24 | 117 | 233 | 284400 | 1600 | 110 |
| 30 | 117 | 256 | 376560 | 1600 | 114 |
| 36 | 117 | 279 | 477000 | 1600 | 118 |
| 42 | 117 | 303 | 586080 | 1600 | 122 |
| 48 | 117 | 328 | 703440 | 1600 | 126 |
| 54 | 117 | 350 | 829440 | 1600 | 131 |
| 60 | 117 | 373 | 963720 | 1600 | 136 |

Scenario \#2. This scenario evaluates the impact of HGL produced by self ignition of stack of four cable trays described in scenario \#1 for location \#11 on the T-L wrap in this location \# 12.

Results

This fire zone is pretty open and location nos. 11 and 12 treely communicate with each other, therefore, the HGL temperatures calculated for scenario \#1 for location \#11 is also assumed to apply here. It should be noted that the bottom cable tray, the auto-Ignition source/combustible for that scenario, is at 12 ft . below the celling and the T-L wrap here is at about 8 ft . below the ceiling, therefore, the target is in the HGL. region of scenario \#1 for location \#11. The resilts are presented in Table 6 below.

Table 6. Temperature-Time History for Fire Scenario 2, Location No. 12.

| Time | HRR io | HRR to | QTOT | iemp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | RGL |  |  |  |
| (min.) | $($ Btu/s) | $($ Btu/s $)$ | (Btu) | Target-(F) | HGL-(F) |
| 0 | 1047 | 1047 | 26 | 100 | 100 |
| 6 | 1234 | 1234 | 444240 | 111 | 111 |
| 12 | 1421 | 1421 | 955800 | 124 | 124 |
| 18 | 1608 | 1608 | 1534580 | 138 | 138 |
| 24 | 1795 | 1795 | 2180880 | 155 | 155 |
| 30 | 1982 | 1982 | 2894400 | 174 | 174 |
| 36 | 2169 | 2169 | 3675240 | 196 | 196 |
| 42 | 2356 | 2356 | 4523400 | 221 | 221 |
| 48 | 2543 | 2543 | 5438880 | 248 | 248 |
| 54 | 2730 | 2730 | 6421680 | 279 | 279 |
| 60 | 2917 | 2917 | 7471800 | 313 | 313 |

Scenario \#3. The ignition source/combustible considered in this scenario is maintenance refuse being left in a plastic container on the floor against the drywell, placing the T-L. wrap directly "in-plume" above. It was assumed that 50 lbm of maintenance trash (rags, paper, etc.) were left in the plastic container, for a total Q available of $800,000 \mathrm{BTUs}(400,000 \mathrm{BTUs}$ for
the trash, and 400,000 BTUs for the plastic can, Ref. 6, page E2-1). The heat release rate is 138 BTUs/s (Ref. 1, Appendix I-E).

Results

As shown in Table 7, the "in-plume" temperatures (ir.cluding the HGL energy contribution) for the T-L wrap target do not exceed $284^{\circ} \mathrm{F}$ by the time this scenario for the transient fuel source is ended. Therefore, the T-L wrap is not damaged. A sample model worksheet is provided in Appendix A, Figure A-5.

Table 7. Temperature-Time History for Fire Scenario 3, Location No. 12.

| Time | HRR to | HRR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL |  |  |  |
| (min.) | (Btu/s) | (Btu/s) | (Btu) | Target - (F) | HGL - (F) |
| 0 | 138 | 138 | 0 | 277 | 100 |
| 6 | 138 | 138 | 49880 | 278 | 101 |
| 12 | 138 | 138 | 99360 | 279 | 101 |
| 18 | 138 | 138 | 149040 | 279 | 102 |
| 24 | 138 | 138 | 198720 | 280 | 103 |
| 30 | 138 | 138 | 248400 | 281 | 10.5 |
| 36 | 138 | 138 | 298080 | 281 | 104 |
| 42 | 138 | 138 | 347760 | 282 | 105 |
| 48 | 138 | 138 | 397440 | 282 | 105 |
| 54 | 138 | 138 | 447120 | 283 | 106 |
| 60 | 138 | 138 | 496800 | 284 | 107 |

## Location No. 9

There are two vertical T-L. wrapped conduits by the stairwell at this location. During the walkdowns, no fixed combustibies were noted within a 15 '-radius. Possibility of HGL entrapment was also discarded during walkdowns. There is an extension cord within 2-3 ft. of these T-L conduit runs but its impact is deeined insignificant. There are no pumps nearby and it is very unlikely that any combustibles will be stored in this area. No scenarios were developed for this location.

## Location No. 15

This area "the TIP room" was not accessible during the walkdowns due to high radiation. The information gathered here was obtained from Plant personnel and drawings. There are two TL wrapped penetrations at this location (Penetration Nos. 44 \& 54, see drawing no. B\&R E1317). The dimensions for this room were estimated at $203 / 4 \mathrm{ft}$. by 20 ft . by 15 ft . height (from general arrangement drawings). Since there are no maintenance activities being done in this area (High Radiation area) during power operation, only one transient ignition source was modeled for this room.

Scenario \#1. A wooden ladder was noted stored in this area from the pictures obtained prior to the application of the thermo-lag fire barrier for this location (pictures from Susan Hopson from OC PLant Rad. Engineering). This is modeled here as a transient ignition source/combustible. The heat release rate considered for this source (wooden ladder) is 12 BTU/sec- ft2 (Ref. 2, FIVE Table 2E); it is assumed that a total surface area of 10 ft 2 is burning, the total HRR for this source is, therefore, $120 \mathrm{BTU} / \mathrm{sec}$. The total available heat (Otot) is 400,000 BTUs [it assumes the ladder weighs $50 \mathrm{lbra} \times 8,000 \mathrm{BTU} / \mathrm{lbm}$ (per Ref. 6, Page E2-3)]. The ignition source was modeled conservatively "against the wall". The T-L wrapped penetrations were assumed "in-plume" located about half-way beiow the ceiling at 7.5 ft . above the fire source.

Rezults

The analysis as shown Table 8 indicates that as a result of the ladder igniting, the "in-plume" temperatures in the room reach $863^{\circ} \mathrm{F}$ at one hour into the scenario. Therefore, the T-L wrap is not damaged. A sample model worksheet is provided in Appendix A, Figure A-6.

Table 8. Temperature-Time History for Fire Scenario 1, Location No. 15.

| Time | HRTR to | HRRA to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL |  |  |  |
| (min.) | (Btu/s) | (Etu/s) | (3tu) | Target - (F) | HGL - (F) |
| 0 | 120 | 120 | 0 | 557 | 100 |
| 6 | 120 | 120 | 43200 | 582 | 125 |
| 12 | 120 | 120 | 86400 | 608 | 151 |
| 18 | 120 | 120 | 129600 | 635 | 178 |
| 24 | 120 | 120 | 172800 | 664 | 207 |
| 30 | 120 | 120 | 216000 | 693 | 236 |
| 36 | 120 | 120 | 259200 | 724 | 267 |
| 42 | 120 | 120 | 302400 | 757 | 300 |
| 48 | 120 | 120 | 345600 | 791 | 334 |
| 54 | 120 | 120 | 388800 | 826 | 369 |
| 60 | 120 | 12. | 432000 | 863 | 406 |

## 5. FIRE MODELING SUMMARY

Fire modeling was performed for three locations containing Thermo-lag wrap in Oyster Creek fire zone RB-FZ-1E. The analysis results are presented as temperature-time history profiles in Tables 1 through 8 . These results are used in the following section to develop Total Heat Load and then convert to equivaient ASTM E-119 exposures.

## 6. APPROXIMATING EXPOSURE TIME-HISTORY OF THERMO-LAG SUBCONFIGURATIONS - THE TOTAL HEAT LOAD CONCEPT

Calculation of an exposure time-temperature profile for a specfic T-L Sub-configuration is based on the methodology presented in Reference 1. The time -temperature profiles are used to develop the total heat load (Reference 1, Appandix Ill-A) for each fire exposure. The total heat load concept holds that the area under the incident heat flux vs. timn curve, or total heat load, can be used as a measure of fire severity. Comparing the total heat load of a fire
scenario exposure to the total heat load of a test exposure (ASTM E-118) allows one to predict the response of a Thenno-Lag barrier to the scenario axposure. The total heat load per unit area of fire barrier surface is equal to the area under the incident heat flux-time curve, corrected for convection effects as described in Reference 1, Appendix ill-A.

### 6.1 Total Heat Load Results

| Locstion <br> No. (GA <br> DWG, <br> App. D) | Scenario | Description | Peak <br> Tsmperature <br> at Target <br> (FF) | Total Reat Losd <br> Equivalent E- <br> 119 Exposure <br> Duration (min.) |
| :--- | :--- | :--- | :--- | :--- |
| 10 | 1 | OlTransient | 805 | $<10$ |
| 10 | 2 | Cable Tray | 313 | $<12$ |
| 11 | 1 | Cable tray | 1600 | $>60$ |
| 11 | 2 | Cable Tray | Radiant <br> Exposure | $>1.2 \mathrm{Ft}$. |
| 11 | 3 | CS Booster Pump | 135 | $<10$ |
| 12 | 1 | Cable Tray | 1600 | $>60$ |
| 12 | 2 | Cable Tray | 313 | $<12$ |
| 12 | 3 | Maintenance Reluse | 284 | $<18$ |
| 9 | None |  |  |  |
| 15 | 1 | Wooden Ladder | 863 | $<24$ |

Appendix C contains tables with Total Heat Load values calculated for Thermo-Lag scenarios.

## 7. REFERENCES

1. EPRI Report "Methods for Evaluation of Cable Wrap Fire Barrier Performance", July, 1995
2. EPRI TR-100370, "Fire-Induced Vulnerability Evaluation," April 1992.
3. GPUN, "Fire Hazards Analysis Report - OC," Doc. No. 990-1746, Rev. 8, May 24, 1995.
4. GPUN - OC Procedure No. 120.5 (Revision 5), May 30, 1993.
5. Thermo-Lag 330-1 Combustibility Study, NEI, 1994.
6. GPUN - TMI-1 Procedure No. 1035 (Revision 24), August 23, 1994.
7. Personal communication with Malcom Gonzales of GPUN, in charge of plant oll program.

## Appendlb: A

Sample Scenario Worksheets

Figure A-1

| Temperature calculation at time |  | min | 1 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | $f 2$ | 12140 |
| 1 | Location of Target |  | Plume |
| 2 | Height of Target above Fire Source | f | 12.00 |
| 3 | Height from Fire Source to Celiing, H | f | 25.00 |
| 4 | Ratio of Target Height/Ceiling Height |  | 0.480 |
| 5 | Longitudinal Distance from Fire Source to Target, L | f | NA |
| 8 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | $f$ | NA |
| 8 | Height to Width Ratio, H/W |  | NA |
| 8 | Peak Fire Intensity | Biu/s | 1485 |
| 10 | Fire Location Factor |  | 1 |
| 11 | Effective Fire Intensity | Btu/s | 1485 |
| 12 | Plume Temperature Rise at Target | F | 704 |
| 13 | Temp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | $F$ | 704 |
| 15 | Temperature at Target | $F$ | 805 |
| 16 | Temperature in HGL | F | 101 |
| 17 | Qtot to HGL | Btu | 89100 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 18 | Calculated Onet | Btu | 5348 |
| 20 | Calculated Enclosure Volume, V | $f 13$ | 303500 |
| 21 | Calculated QnelV | Qnet/th | 0.02 |
| 22 | HGL. Temperature increase | F | 1 |
|  |  |  |  |

Figure A-2

| Temperature caiculation at time |  | min | 6 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | 12 | 12140 |
| 1 | Location of Target |  | Plume |
| 2 | Height of Target above Fire Source | $f$ | 9.00 |
| 3 | Height from Fire Source to Ceiling, H | $f$ | 12.00 |
| 4 | Ratio of Target Height/Ceiling Height |  | 0.750 |
| 5 | Longitudinal Distance from Fire Source to Target, L | กิ | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | $f$ | NA |
| 8 | Height to Width Ratio, HWW |  | NA |
| 8 | Peak Fire Intensity | Btu/s | 1123 |
| 10 | Fire Lncation Factor |  | 1 |
| 11 | Effective Fire intensity | Btws | 1123 |
| 12 | Plume Temperature Rise at Target | F | 943 |
| 13 | Temp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | F | 943 |
| 15 | Temperature at Target | $F$ | 1054 |
| 16 | Temperature in HGL | $F$ | 111 |
| 17 | Qtot to HGL | Btu | 444240 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated Qnet | Btu | 26654 |
| 20 | Calculated Enclosure Volume, V | $f 13$ | 145680 |
| 21 | Calculated Qnet/V | Qnet/f3 | 0.18 |
| 22 | HGL Temperature Increase | $F$ | 11 |
|  |  |  |  |

## WORKSHEET 3: RADIANT EXPOSURE SCENARIOS

| 1 | Critical Radiant Flux to the Target (Representative value $=1$ ) | Btws/fl2 | 2.2 | <-User Input <br> <-User Input |
| :---: | :---: | :---: | :---: | :---: |
| 2 | Peak Fire Intensity | Btu/s | 94 |  |
| 3 | Radiant Fraction of Heat Release (Representative value $=0.4$ ) |  | 0.4 | <-User Input |
| 4. | Radiant Heat Release Rate | Btu/s | 37.6 |  |
| 5 | Critical Radiant Flux Distance | $f$ | 1.2 |  |
| If the eqposure fre is bocated wittin thise distance (indicated in Bax 5, of the targot, critical conditions can occur. Outaside this renge, ertical condtions are not indicated for the sconnario under consaliceration. |  |  |  |  |

Figure A-3

| Temperature calculation at time |  | min | 5 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | 12 | 12140 |
| 1 | Location of Target |  | Ceiling Jet |
| 2 | Height of Target above Fire Source | $f$ | 25.00 |
| 3 | Height from Fire Source to Ceiling, H | f | 25.00 |
| 4 | Ratio of Target Height/Ceiling Height | . | 1.000 |
| 5 | Longitudinal Distance from Fire Source to Target, L | $f$ | 15 |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | 0.60 |
| 7 | Enclosure Width, W | fi | 25 |
| 8 | Height to Width Ratio, HWW |  | 8.00 |
| 9 | Peak Fire Intensity | Btu/s | 440 |
| 10 | Fire Location Factor |  | 1 |
| 11 | Effective Fire Intensity | Btw/s | 440 |
| 12 | Plume Temperature Rise at Target | F | 82 |
| 13 | Temp Rise Factor at Target |  | 0.34 |
| 14 | Temperature Rise at Target | $F$ | 31 |
| 15 | Temperature at Target | F | 132 |
| 16 | Temperature in HGL | F | 102 |
| 17 | Qtot to HGL | Btu | 132000 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated Qnet | Btu | 7920 |
| 20 | Calculated Enclosure Volume, V | $f 13$ | 303500 |
| 21 | Calculated Qnetw | Qnev/it ${ }^{\text {a }}$ | 0.03 |
| 22 | HGL Temperature Increase | $F$ | 2 |
|  |  |  |  |

Figure A-4

| Temperature calculation at time |  | min | 6 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | 12 | 12140 |
| 1 | Location of Target |  | Piume |
| 2 | Height of Target above Fire Source | \% | 0.10 |
| 3 | Height from Fire Source to Ceiling, H | fi | 8.00 |
| 4 | Ratio of Target Height/Ceiling Height | . | 0.013 |
| 5 | Longitudinal Distance from Fire Source to Target, L | ה | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | $f$ | NA |
| 8 | Height to Width Ratio, HWW |  | NA |
| 9 | Peak Fire Intensity | Btw/s | 117 |
| 10 | Fire Location Factor |  | 2 |
| 11 | Effective Fire Intensity | Btw/s | 234 |
| 12 | Plume Temperature Rise at Target | F | 1600 |
| 13 | Temp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | $F$ | 1600 |
| 15 | Temperature at Target | F | 1600 |
| 16 | Temperature in HGL | F | 102 |
| 17 | Qtot to HGL | Btu | 58320 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 18 | Calculated Qnet | Btu | 3499 |
| 20 | Calculated Enclosure Volume, V | $f 13$ | 97120 |
| 21 | Calculated Qnet/V | Qnet/fi3 | 0.04 |
| 22 | HGL Temperature Increase | F | 2 |
|  |  |  |  |

Figure A-5

| Temperature calculation at time |  | min | 6 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | 12 | 12140 |
| 1 | Location of Target |  | Plume |
| 2 | Height of Target above Fire Source | fi | 14.00 |
| 3 | Height from Fire Source to Ceiling. H | ก | 22.00 |
| 4 | Ratio of Target Height/Ceiling Height |  | 0.636 |
| 5 | Longitudinal Distance from Fire Source to Target, L | fi | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | fif | NA |
| 8 | Height to Width Ratio, HWW |  | NA |
| 9 | Peak Fire Intensity | Btw/s | 138 |
| 10 | Fire Location Factor |  | 2 |
| 11 | Effective Fire Intensity | Btu/s | 276 |
| 12 | Plume Temperature Rise at Target | F | 177 |
| 13 | Temp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | F | 177 |
| 15 | Temperature at Target | $F$ | 278 |
| 16 | Temperature in HGL | F | 101 |
| 17 | Qtot to HGL | Btu | 49680 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 18 | Calculated Qnet | Btu | 2981 |
| 20 | Calculated Enclosure Volume, V | $f 13$ | 267080 |
| 21 | Calculated Qnet/V | Qnetifl | 0.01 |
| 22 | HGL Temperature Increase | $F$ | 1 |
|  |  |  |  |

Figure A-6

| Temperature calculation at time |  | min | 6 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | $f 12$ | 415 |
| 1 | Location of Target |  | Plume |
| 2 | Height of Target above Fire Source | fi | 7.50 |
| 3 | Height from Fire Source to Ceiling, H | f | 15.00 |
| 4 | Ratio of Target Height/Ceiling Height |  | 0.500 |
| 5 | Longitudinal Distance from Fire Source to Target, L | f | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | fi | NA |
| 8 | Height to Width Ratio, H/W |  | NA |
| 9 | Peak Fire Intensity | Btu/s | 120 |
| 10 | Fire Location Factor |  | 2 |
| 11 | Effective Fire Intensity | Btu/s | 240 |
| 12 | Plume Temperature Rise at Target | F | 457 |
| 13 | Temp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | F | 457 |
| 15 | Temperature at Target | F | 882 |
| 16 | Temperature in HGL | $F$ | 125 |
| 17 | Qiot to HGL | Btu | 43200 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated Qnet | Btu | 2592 |
| 20 | Calculated Enclosure Volume, V | $f 13$ | 6225 |
| 21 | Calculated Qnet/V | Qnet/f3 | 0.42 |
| 22 | HGL Temperature Increase | $F$ | 25 |
|  |  |  |  |

## Appendix B

Basis for Following ASTM E-119 Time-Temperature Relationship When T(HCL) is $1000^{\circ} \mathrm{F}$

# TRANSITION TO <br> ASTM E-119 TIME-HISTORY <br> IN THE HOT GAS LAYER 

ISSUE:

The Fire Hazard Tool currently under development in the EPRI Cable Wrap Fire Esrrier Tailored Coliaboration assumes that the fire compartment temperature will follow the ASTM E119 time-temperature relationship as soon as the compartment hot gas layer reaches $1000^{\circ} \mathrm{F}$.

## BASIS:

The Handbook (Ref., pg. 6-75) states that "Flashover of an enclosure is likely to occur if the temperature of the upper gas layer reaches approximately $1000^{\circ} \mathrm{F}\left(600^{\circ} \mathrm{C}\right)$.

The EPRI assumption of $1000^{\circ} \mathrm{F}$ is therefore conservative.

As stated near the top of page 6-67 of the Handbook, the intensity of the fire will be somewhat lower when the walls and ceiling absorb significant amounts of energy rather than when act as insulation or radiation barriers.

Since power plant compartments are typically concrete barriers which absorb significant amounts of heat, the intensity of fire will tend to be less severe than fires in compariments with insulated barriers.

ASTM E-119 has been the accepted standard for testing fire barrici $s /$ stems since the early 1900's. The standard, which was developed from actual fire tests, has been used to evaluate the fire endurance capabilities of fire barriers since it was developed. The timetemperature relationship represented by the curve represents a fully-involved fire compartment.

According to Fred Mowrer of the University of Maryland, plume and ceiling effects disappear in a fully involved fire compartment as the plume/ceiling jet gas are no longer more buoyant than surrounding gases.

Use of the ASTM E-119 time-temperature relationship to represent a fully involved power plant compartment fire is therefore consistent with industry practice. Assuming that that occurs when the hot gas layer reaches $1000^{\circ} \mathrm{F}$ is more conservative than the $1100^{\circ} \mathrm{F}$ noted in the Mandbook.

Reference: NFPA Fire Protection Handbook, Seventeenth Edition, Section 6/Chapter 6.

Appendix C

## Total Heat Load Results

Figure $\mathrm{C}_{-1}$

7 otal Heat Load
iscenario 1, Loc. No. 10

|  |  |  | Target | Hot Gas Layer | ASTM E-118 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| (mili) | (F) | (Btu/s-sf) | (1000Btu/s) | (1000Btu/sf) | (1000Btu/sf) |
| 0 | 804 | 1.6 | 0.0 | 0.00 | 0.00 |
| 1 | 805 | 1.7 | 0.10 | 0.02 | 0.00 |
| 2 | 101 | 0.2 | 0.16 | 0.03 | 0.02 |
| 10 | 101 | 0.2 | 0.26 | 0.14 | 1.23 |
| 15 | 101 | 0.2 | 0.33 | 0.21 | 2.80 |
| 20 | 109 | 0.2 | 0.40 | 0.28 | 4.66 |
| 25 | 101 | 0.2 | 0.47 | 0.35 | 6.75 |
| 30 | 101 | 0.2 | 0.54 | 0.41 | 9.04 |
| 40 | 101 | 0.2 | 0.67 | 0.55 | 14.10 |
| 50 | 101 | 0.2 | 0.81 | 0.89 | 19.73 |
| 60 | 101 | 0.2 | 0.85 | 0.82 | 25.84 |

Figure C -2

Total Heat Load
Scenario 1, Loc. No. 11

|  |  |  | Target | Hot Gas Layer | ASTM E-119 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btu/s-sf) | (1000Btu/s) | (1000Btu/s!) | (1000 Btw/sf) |
| 0 | 1043 | 3.0 | 0.0 | 0.00 | 0.00 |
| 6 | 1054 | 3.1 | 1.10 | 0.11 | 0.41 |
| 12 | 1169 | 4.0 | 2.37 | 0.20 | 1.65 |
| 18 | 1281 | 5.1 | 4.01 | 0.30 | 3.56 |
| 24 | 1391 | 6.4 | 6.07 | 0.41 | 5.83 |
| 30 | 1500 | 7.8 | 8.83 | 0.54 | 8.64 |
| 36 | 1600 | 9.5 | 11.76 | 0.68 | 11.59 |
| 42 | 1600 | 9.5 | 15.17 | 0.82 | 14.73 |
| 48 | 1600 | 9.5 | 18.57 | 0.95 | 18.03 |
| 54 | 1600 | 9.5 | 21.98 | 1.11 | 21.49 |
| 60 | 1600 | 9.5 | 25.38 | 1.27 | 25.15 |

Figure $\mathrm{C}-3$

Total Heat Load
Scenario 3,Loc. No. 11

|  |  |  | Target | Hot Gas Layer | ASTM E-918 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| $(\mathrm{min})$ | (F) | (Btu/s-sf) | $(1000$ Btu/sf) | $(1000$ Btw/sf | $(1000$ Btu/st $)$ |
| 0 | 131 | 0.3 | 0.0 | 0.00 | 0.00 |
| 5 | 132 | 0.3 | 0.08 | 0.08 | 0.34 |
| 10 | 134 | 0.3 | 0.17 | 0.16 | 1.37 |
| 12 | 135 | 0.3 | 0.20 | 0.18 | 1.93 |
| 13 | 104 | 0.2 | 0.22 | 0.20 | 2.21 |
| 15 | 104 | 0.2 | 0.25 | 0.23 | 2.84 |
| 20 | 104 | 0.2 | 0.32 | 0.30 | 4.71 |
| 30 | 104 | 0.2 | 0.46 | 0.44 | 8.07 |
| 40 | 104 | 0.2 | 0.60 | 0.58 | 14.73 |
| 50 | 104 | 0.2 | 0.73 | 0.71 | 19.76 |
| 60 | 104 | 0.2 | 0.87 | 0.85 | 25.87 |

Figure C-4

Total Heat Load
Scenario 1, Loc. No. 12

|  |  |  | Target | Hot Gas Layer | ASTM E-118 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btu/s-sf) | $(1000$ Btu/sf | $(1000$ tu/s) | $(1000$ Btu/sf) |
| 0 | 1600 | 8.5 | 0.0 | 0.00 | 0.00 |
| 6 | 1600 | 8.5 | 3.41 | 0.11 | 0.41 |
| 12 | 1600 | 9.5 | 6.82 | 0.19 | 1.65 |
| 18 | 1600 | 9.5 | 10.23 | 0.28 | 3.56 |
| 24 | 1600 | 8.5 | 13.64 | 0.36 | 5.93 |
| 30 | 1600 | 9.5 | 17.05 | 0.45 | 8.64 |
| 36 | 1600 | 9.5 | 20.46 | 0.54 | 11.59 |
| 42 | 1600 | 9.5 | 23.86 | 0.63 | 14.73 |
| 48 | 1600 | 8.5 | 27.27 | 0.73 | 18.03 |
| 54 | 1600 | 9.5 | 30.68 | 0.83 | 21.48 |
| 60 | 1600 | 9.5 | 34.09 | 0.93 | 25.15 |

Figure C-5

Total Heat Load
Scenario 3, Loc. No. 12

|  |  |  | Target | Hot Gas Layer | ASTM E-118 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btu/s-sf) | (1000Btu/s? | (1000Btu/sf) | (1000Btu/sf) |
| 0 | 277 | 0.5 | 0.0 | 0.00 | 0.00 |
| 6 | 278 | 0.5 | 0.17 | 0.11 | 0.41 |
| 12 | 279 | 0.5 | 0.34 | 0.18 | 1.65 |
| 18 | 279 | 0.5 | 0.51 | 0.27 | 3.56 |
| 24 | 280 | 0.5 | 0.68 | 0.35 | 5.93 |
| 30 | 281 | 0.5 | 0.85 | 0.44 | 8.64 |
| 36 | 281 | 0.5 | 1.02 | 0.52 | 11.59 |
| 42 | 282 | 0.5 | 1.19 | 0.60 | 14.73 |
| 48 | 282 | 0.5 | 1.37 | 0.69 | 18.03 |
| 54 | 283 | 0.5 | 1.54 | 0.77 | 21.49 |
| 60 | 284 | 0.5 | 1.71 | 0.86 | 25.15 |

Figure C-6

Total Heat Load
Scenario 1, Loc. No. 15

|  |  |  | Target | Hot Gas L-syer | ASTM E-119 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Hea! Load |
| (min) | (F) | (Btu/s-sf) | $(1000$ Btu/st $)$ | $(1000$ Biu/s! | $(1000$ Btu/s! $)$ |
| 0 | 557 | 0.9 | 0.0 | 0.00 | 0.00 |
| 6 | 582 | 1.0 | 0.34 | 0.11 | 0.41 |
| 12 | 608 | 1.0 | 0.70 | 0.22 | 1.65 |
| 18 | 635 | 1.1 | 1.07 | 0.35 | 3.56 |
| 24 | 664 | 1.2 | 1.48 | 0.47 | 5.93 |
| 30 | 693 | 1.3 | 1.93 | 0.59 | 8.64 |
| 36 | 724 | 1.4 | 2.42 | 0.74 | 11.59 |
| 42 | 757 | 1.5 | 2.93 | 0.82 | 14.73 |
| 48 | 791 | 1.7 | 3.51 | 1.10 | 18.03 |
| 54 | 826 | 1.8 | 4.13 | 1.29 | 21.49 |
| 60 | 863 | 2.0 | 4.80 | 1.49 | 25.15 |

## Appendix D

## General Arrangement Drawings



PLAN ELEV $23^{\prime} \cdot 6^{\circ}$
FIRE ZONE
RB-FZ-1E


## TR 102

APPENDIX C

# Using FIVE Fire Modeling to Approximate Exposure TimeHistory for Thermo-Lag Configurations in OC Fire Zone RB-FZ-1F2 

Letter Report<br>September 26, 1995

## Prepared by: <br> GPUN Risk Analysis Group

Using FIVE Fire Modeling to Approximate Exposure Time-History for Thermo-Lag Configurations

## 1. PURPOSE

The purpose of this report is to document the detalied fire modeling periormed for fire scenarios involving Thermo-Lag wrapped cable condults as identified during plant walkdowns of OC fire zone RB-FZ-1F2. Thermo-Lag exposure time-history results were obtained for each T-L configuration determined to be impacted by a fire source/combustible.

## 2. APPROACH

The approach followed for modeling of fire scenarios is based on methodology detailed in the EPRI "Methods for Evaluation of Cable Wrap Fire Barrier Performance" document (Ref. 1) which was based on fire modeling techniques developed for FIVE (Ref. 2). Fire modeling was accomplished in two steps:

1. Screening walkdowns to eliminate from further consideration those fire ignition sources and potential fire scenarios that cannot develop into damaging fires.
2. Fire modeling of specific scenarios that were not screened out in the first step.

For each Thermo-Lag configuration analyzed, one or more fire scenarios (including bounding scenarios) were postulated based on the following:
a. information gathered during plant walkdowns (i.e., fixed or transient ignition source and Thermo-Lag target location, type and quantity of combustibles present in zone);
b. information obtained from reviewing applicable sections in Appendix R documents for OC (Ref. 3);
c. information obtained from reviewing OC procedure 120.5, Rev. 5, "Control of Combustibles." (Ref. 4)
d. information provided by GPUN engineers (specializing in fire protection, risk assessment, mechanical).

Thermo-Lag FIVE analyses were performed using computerized Excel spreadsheets which duplicate FIVE Worksheets 1, 2, or 3 and are linked to tables containing the same information as in FIVE Reference Tables. For each Thermo-Lag fire modeling scenario, results are presented as exposure time-temperature profiles. in all scenarios evaluated, time-dependent temperatures were derived by incrementally adding the total heat content of the fuel into the fire compartment hot gas layer.

## 3. FIRE MODELING ASSUMPTIONS

The following Thermo-Lag fire modeling assumptions were made:

1. Worst-case fixed fire source scenarios were modaled for each Inenno-Lag subconfiguration. The determination of which fixed fire sjurce might cause the worst-
was based on minimum damage threshold heights and distances calculated for typical fixed fire sources (e.g., electrical cabinets, electrical motors, lube oil in pumps) found at Oyster Creek.
2. Worst-case transient fire source scenarios observed during walkdowns were also modeled. Worst-case transient fire scenarios considered were similar to the type and quantities of transient combustibles allowed by OC station procedure (Ref. 4).
3. To ensure conservative results, worst-case fire plume (or ceiling jet) scenarios and/or radiant flux scenario were first modeled. For T-L sub-configurations not in the plume or celling jet, hot gas layer fire scenarios were evaluated.
4. Electrical cable inside cabinets, motor control centers, switchgear and cable trays was assumed to be non-qualified (i.e., non IEEE-383 type). This assumption is conservative since electrical cable purchased and installed at Oyster Creek in the past decade was IEEE-383 type.
5. The Fire Hazard Tool developed for the EPRI Cable Wrap Fire Barrier Tallored Collaboration assumes that the fire compartment temperature will follow the ASTM E-119 time-temperature curve as soon as the compartment hot gas layer (HGL) reaches $1000^{\circ} \mathrm{F}$. When the HGL temperature reaches $1000^{\circ} \mathrm{F}$, the plume (or ceiling jet) temperatures are assumed to follow that of the E-119 (see discussion in Appendix B).
6. The Thermo-Lag wrap is assumed to burn by tiseff (i.e., even after the initial fire source is out of fuel) unless otherwise stated. Per, the EPRI Method (Ref. 1, Appendix I-D), credit may be taken for a time lag before the Thermo-Lag ignites based on exposure temperature.
7. The maximum plume and ceiling jet temperature is assumed to be $1600^{\circ} \mathrm{F}$, which corresponds to flame temperature. This assumption is valid until hot gas layer temperature exceeds $1600^{\circ} \mathrm{F}$, at which point the plume temperature is assumed to be equal to hot gas layer temperature.

## 4. FIRE MODELING RESULTS

### 4.1 Fire Zone RB-FZ-1F2

Fire zone RB-FZ-1F2 is the Reactor Building South-West Comer Room at Elevation-19'6'. This fire zone houses two core spray pumps, the reactor building equipment drain tank and, the reactor building equipment drain pump. The T-L wrap conduit run in this fire zone starts at Elevation 2 ft . (about 21 ft . off the floor) and runs along the west wall and then curves and exits through the ceiling. The general location of the T-L wrap in this fire zone is shown as location \# 2 on the general arrangement drawing shown in Appendix D. This area was not accessible during the walkdowns due to high radiation and the information used here was obtained from previously taken pictures, drawings, and communication with plant fire protection personnel.

## Scenario No. 1

The fixed fire ignition source considered here is the Reactor Building Equipment Drain Pump ( P $\mathbf{2 2 - 0 0 1 )}$. This pump contains up to 2.5 gallons of lube oll in the motor and the gearbox (Ref. 6) and sits at -6 ' Elevation (General Arrangement Drawings). This source was chosen because it is more bounding than the core spray pumps. Each of core spray pumps contain up to 2 gallons of oil (Ref. 6) and they are at Elevation -19 ft . The total Q avallable for this source is $387,600 \mathrm{BTU}$ (Ref. 5, Page E2-1, assumed to be the same at OC), heat release rate $110 \mathrm{BTUs} / \mathrm{sec}-\mathrm{ft}$ (from Ref. 2, Table 2E). The potential oll spread area around the pump was estimated at $5 \mathrm{ft}^{2}$ The Thermo-Lag wrap is located 8 ft above the fire source at its closest proximity and is outside of fire plume. The floor area for this fire zone is $560 \mathrm{ft}^{2}$ (Ref. 3). The ceiling height is 40 ft . (general arrangement drawings). Ambient temperature assumed was $100^{\circ} \mathrm{F}$.

## Results

The time temperature profile scenario calculated by the model are presented in Table 1. The model results indicate that hot gas layer effects in the zone due to this postulated fire cause the temperature to rise to approximately $194^{\circ} \mathrm{F}$. A sample model worksheet is provided in Appendix A, Figure A-1.

Table 1. Temperature-Time History for Fire Scenario 1.

| Time | HRR ${ }^{\text {to }}$ | HRR' to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL |  |  |  |
| (min.) | (Btu/s) | (Btu/s) | (Btu) | Target - (F) | HGL - (F) |
| 0 | 550 | 550 | 0 | 100 | 100 |
| 5 | 550 | 550 | 165000 | 138 | 138 |
| 10 | 550 | 550 | 330000 | 179 | 179 |
| 11.7 | 550 | 550 | 386100 | 194 | 194 |
| 12 | 0 | 0 | 386100 | 194 | 194 |
| 15 | 0 | 0 | 386100 | 194 | 194 |
| 20 | 0 | 0 | 386100 | 194 | 194 |
| 30 | 0 | 0 | 386100 | 194 | 194 |
| 40 | 0 | 0 | 386100 | 194 | 194 |
| 50 | 0 | 0 | 386100 | 194 | 194 |
| 60 | 0 | 0 | 386100 | 194 | 134 |

## Scenario No. 2

This fire scenario modeled a transient ignition source/combustible, a 5 gallon container containing lube oil. Per Oyster Creek Procedure "Control of Combustibles," Procedure No. 120.5, Rev. 4 (Ref. 4), combustible liquids could be transported throughout the plant in an approved closed metal container (but not necessarily a safety-can). For this fire scenario, it was postulated that 1 quart of lube oil is spilied out of a can on the floor and is not cleaned up. The oil is later ignited by a welder working nearby or another transient ignition source and begins to burn. The oil fire could potentially impact a T-L. wrap 20 feet above the floor.

The heat release rate for the lube oil is $110 \mathrm{Btu} / \mathrm{s}-\mathrm{ft}^{2}$ (from Ref. 2, Table 2E). The oil could spread unconfined over an area of $13.5 \mathrm{ft}^{.2}$ ( $54 \mathrm{ft}^{2} / \mathrm{gal}$ from Ref. 2, Table 3E). The heat release rate assumed for this fire scenario is $1,485 \mathrm{Btu} / \mathrm{s}\left(110 \mathrm{Btu} / \mathrm{s}-\mathrm{ft}^{2} \times 13.5 \mathrm{tt}^{2}\right)$. The heat content of one quart of oil is ( 17,111 Btu/b. * $60 \mathrm{lb} . / 1 \mathrm{ft}^{3}$. * $1 \mathrm{ft} .^{3} / 30$ quarts) 34,222 Btu (from Ref. 2, Table 2 E ). The oil fire duration will be less than one minute, but one minute will be used for conservatism. The virtual surface of the fire is at the floor level. The location factor for this source is one.

## Results

The short oil fire does not cause an exposure temperature high enough to ignite the ThermoLag. The fire is of a short duration, with very slight hot gas layer effects. A sample model worksheet is provided as Appendix A, Figure A-2.

Table 2. Temperature-Time History for Fire Scenario 2.

| Time | HRR to | HRR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL |  |  |  |
| (min.) | (Btu/s) | (Btu/s) | (Btu) | Target - (F) | HGL- (F) |
| 0 | 1485 | 1485 | 0 | 400 | 100 |
| 1 | 1485 | 1485 | 89100 | 414 | 114 |
| 2 | 0 | 0 | 89100 | 114 | 114 |
| 10 | 0 | 0 | 89100 | 114 | 114 |
| 15 | 0 | 0 | 89100 | 114 | 114 |
| 25 | 0 | 0 | 89100 | 114 | 114 |
| 30 | 0 | 0 | 89100 | 114 | 114 |
| 35 | 0 | 0 | 89100 | 114 | 114 |
| 40 | 0 | 0 | 89100 | 114 | 114 |
| 50 | 0 | 0 | 89100 | 114 | 114 |
| 60 | 0 | 0 | 89100 | 114 | 114 |

## 5. FIRE MODELING SUMMARY

Fire modeling was performed for one fixed and one transient fire Source-Thermo-lag wrap configuration in Oyster Creek Fire Zone RB-FZ-1F2. The analysis results are presented as time-temperature profiles. These "esults are used in the following section to develop total heat load and then convert to equivi'ent ASTM E-119 exposures.

## 6. APPROXIMATING EXPOSURE TIME-HISTORY OF THERMO-LAG SUBCONFIGURATIONS - THE TOTAL HEAT LOAD CONCEPT

Calculation of an exposure time-temperature profile for a specific T-L Sub-configuration is based on the methodology presented in Reference 1. The time -temperature profiles are used to develop the total heat load (Reference 1, Appendix III-A) for each fire exposure. The total heat load concept holds that the area under the incident heat flux vs. time curve, or total heat load, can be used as a measure of fire severity. Comparing the total heat load of a fire scenario exposure to the total heat load cif a test exposure (ASTM E-119) allows one to predict the response of a Thermo-Lag barrier to the scenario exposure. The total heat load per unit area of fire barrier surface is equal to the area under the incident heat flux-time curve, corrected for convection effects as described in Reference 1, Appendix II-A.

### 6.1 Total Heat Load Results

| Location <br> No.(GA <br> DWG, <br> App. D) | Scenario | Description | Peak <br> Temperature at <br> Target ( | Total Heat Load <br> Equivalent E-119 <br> Exposure Duration <br> (min.) |
| :--- | :--- | :--- | :--- | :--- |
| 2 | 1 | Oilfire at RBED <br> Pump (P-22-001) | 134 | $<12$ |
| 2 | 2 | OllTransient | 414 | $<10$ |

Appendix C contains tables with Total Heat Load values calculated for Thermo-Lag scenarios 1 , and 2.

## 7. REFERENCES

1. EPRI Report "Methods for Evaluation of Cable Wrap Fire Barrier Performance", July, 1995
2. EPRI TR-100370, "Fire-Induced Vulnerability Evaluation," April 1992.
3. GPUN, "Fire Hazards Analysis Report - OC," Doc. No. 990-1746, Rev. 8, May 24, 1995.
4. GPUN - OC Procedure No. 120.5 (Revision 5), May 20, 1993.
5. GPUN - TMI-1 Procedure No. 1035 (Revision 24), August 23, 1994.
6. Personal communication with Malcolm Gonzales of GPUN, in charge of plant oil program.

Appendix A
Sample Scenario Worksheets

Figure A-1

| Temperature caiculation at time |  | min | 5 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | $f 12$ | 560 |
| 1 | Location of Target |  | Hot Gas Layer |
| 2 | Height of Target above Fire Source | ft | 8.00 |
| 3 | Height from Fire Source to Ceiling. H | $f$ | 28.00 |
| 4 | Ratio of Target Height/Celing Height |  | 0.286 |
| 5 | Longitudinal rjistance from Fire Source to Target, l | fi | NA |
| 6 | Longitudiral Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | f | NA |
| 8 | Height to Width Ratio, H/W |  | NA |
| 8 | Peak Fire Intensity | Btwis | 550 |
| 10 | Fire Location Factor |  | 1 |
| 11 | Effective Fire Intensity | Btu/s | 550 |
| 12 | Plume Temperature Rise at Target | F | 713 |
| 13 | Temp Rise Factor at Target |  | 0.00 |
| 14 | Temperature Rise at Target | F | 0 |
| 15 | Temperature at Target | F | 138 |
| 16 | Temperature in HGL | $F$ | 138 |
| 17 | Qtot to HGL | Btu | 165000 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Caiculated Qnet | Btu | 9900 |
| 20 | Calculated Enclosure Volume, V | $f 13$ | 15680 |
| 21 | Calculated Onet/V | Qnet/f3 | 0.63 |
| 22 | HGL Temperature increase | F | 38 |
|  |  |  |  |

Figure A-2

| Temperature calculation at time |  | min | 1 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Fioor Area |  | $f 2$ | 560 |
| 1 | Location of Target |  | Plume |
| 2 | Height of Target above Fire Source | $f$ | 20.00 |
| 3 | Height from Fire Source to Ceiling, H | $f$ | 40.00 |
| 4 | Ratio of Target Heighuceiling Height |  | 0.500 |
| 5. | Longitudinal Distance from Fire Source to Target, L | f | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | $f$ | NA |
| 8 | Height to Width Ratio, HWW |  | NA |
| 9 | Peak Fire Intensity | Btu/s | 1485 |
| 10 | Fire Location Factor |  | 1 |
| 11 | Effective Fire Intensity | Btw/s | 1485 |
| 12 | Plume Temperature Rise at Target | F | 300 |
| 13 | Ternp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | $F$ | 300 |
| 15 | Temperature at Target | F | 414 |
| 16 | Temperature in HGL | $F$ | 114 |
| 17 | Qtot to HGL. | Btu | 89100 |
| 18 | Estimated Heat Loss Fraction |  | 0.84 |
| 19 | Calculated Qnet | Btu | 5348 \% |
| 20 | Caiculated Enclosure Volume, V | $f t 3$ | 22400 |
| 21 | Calculated Qnet/V | Qnet/73 | 0.24 |
| 22 | HGL. Temperature Increase | F | 14 |
|  |  |  |  |

## Appendix B

Basis for Following ASTM E-119 Time-Temperature Relationship When T(HGL) is $1000{ }^{\circ} \mathrm{F}$

# TRANSITION TO <br> ASTM E-119 TIME-HISTORY <br> IN THE HOT GAS LAYER 

## ISSUE:

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## BASIS:

The Handbook (Ref., pg. 6-75) states that "Flashover of an enclosure is likely to occur if the temperature of the upper gas layer reaches approximately $1000^{\circ} \mathrm{F}\left(600^{\circ} \mathrm{C}\right)$.

The EPRI assumption of $1000^{\circ} \mathrm{F}$ is therefore conservative.

As stated near the top of page 6-67 of the Handbook, the intensity of the fire will be somewhat lower when the walis and ceiling absorb significant amounts of energy rather than when act as insulation or radiation barriers.

Since power plant compartments are typically concrete barriers which absorb significant amounts of heat, the intensity of fire will tend to be less severe than fires in compartments with insulated barriers.

ASTM E-119 has been the accepted standard for testing fire barrier systems since the early 1900's. The standard, which was developed from actual fire tests, has been used to evaluate the fire endurance capabilities of fire barriers since it was developed. The timetemperature relationship represented by the curve represents a fully-involved fire compartment.

According to Fred Mowrer of the University of Maryland, plume and ceiling effects disappear in a fully involved fire compartment as the plume/ceiling jet gas are no longer more buoyant than surrounding gases.

Use of the ASTM E-119 time-temperature relationship to represent a fully involved power plant compartment fire is therefore consistent with industry practice. Assuming that that occurs when the hot gas layer reaches $1000^{\circ} \mathrm{F}$ is more conservative than the $1100^{\circ} \mathrm{F}$ noted in the Handbook.

Reference: NFPA Fire Protection Handbook, Seventeenth Edition, Section 6/Chapter 6.

Appendix C

## Total Heat Load Results

Figure C-1

Total Heat Load

| Scenario 1 |
| :--- |
|    Target Hot Gas Layer ASTM E- 119 <br> Time Temperature Inc Heat Flux Total Heat Load Total Heat Load Total Heat Load <br> $(\mathrm{min})$ $\left(F_{1}\right.$ (Btu/s-sf) $(1000$ Btw/sf) $(1000$ Btw/s? $(1000$ Btu/s) <br> 0 100 0.4 0.0 0.00 0.00 <br> 5 138 0.3 0.10 -0.10 0.34 <br> 10 178 0.4 0.20 0.20 1.37 <br> 11.7 194 0.4 0.24 0.24 1.85 <br> 12 194 0.4 0.25 0.25 1.93 <br> 15 194 0.4 0.32 0.32 2.88 <br> 20 194 0.4 0.45 0.45 4.74 <br> 30 194 0.4 0.70 0.70 9.11 <br> 40 184 0.4 0.95 0.95 14.17 <br> 50 194 0.4 1.20 1.20 19.80 <br> 60 194 0.4 1.45 1.45 25.91 |

Figure C-2

Total Heat Load
Scenario 2

|  |  |  | Target | Hot Gas Layer | ASTM E-118 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btu/s-5!) | (1000Btu/s) | (10008tu/sf) | (1000b'4/s? |
| 0 | 400 | 0.7 | 0.0 | 0.00 | 0.00 |
| 1 | 414 | 0.6 | 0.04 | 0.02 | 0.00 |
| 2 | 114 | 0.2 | 0.06 | 0.03 | 0.02 |
| 10 | 114 | 0.2 | 0.18 | 0.15 | 1.23 |
| 15 | 114 | 0.2 | 0.26 | 0.23 | 2.80 |
| 20 | 114 | 0.2 | 0.33 | 0.30 | 4.66 |
| 25 | 114 | 0.2 | 0.41 | 0.38 | 6.75 |
| 30 | 114 | 0.2 | 0.48 | 0.45 | 9.04 |
| 40 | 114 | 0.2 | 0.63 | 0.60 | 14.10 |
| 50 | 114 | 0.2 | 0.78 | 0.75 | 18.73 |
| 60 | 114 | 0.2 | 0.93 | 0.90 | 25.84 |

## Appendix D

## General Arrangement Drawings



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& \text { TR } 102 \\
& \text { APPEND } \times D
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# Evaluation of Thermo-Lag Cable Wrap Fire Barriers in Oyster Creek Fire Zone TB-FZ-11D 

Report No. 01-0082-05-2965-700-1
July 21, 1995

Prepared by:
Science Applications International Corporation, Inc.

## 1. PURPOSE

The purpose of this report is to document the detailed fire modeling performed for fire scenarios involving Thermo-Lag wrapped cable conduits, as identified during plant walkdowns of the Oyster Creek Turbine Building Basement Floor, South End, Fire Zone TB-FZ-11D. This fire zone includes two separate Thermo-Lag (T-L) conduit configurations (based on plant walkdown notes of October 27, 1994 and Oyster Creek plant drawings of Appendix R Raceway-Turbine Building Basement, Ref. 1). Fire exposure time-temperature profile results were obtained for each of these two T-L configurations, as analyzed in seven fire modeling scenarios involving fixed and transient fire sources.

## 2. APPROACH

The approach followed for modeling of fire scenarios is based on methodology detailed in the EPRI "Methods for Evaluation of Cable Wrap Fire Barrier Performance" document (Ref. 2). which was based on fire modeling techniques developed for FIVE (Ref. 3). Fire modeling was accomplished in two steps:

1. Screening walkdowns to eliminate from further consideration those fire ignition sources and potential fire scenarios that can't develop into damaging fires.
2. Fire modeling of specific scenarios that were not screened out in the first step.

For each of the two Thermo-Lag configuration analyzed, one or more fire scenarios (including bounding scenarios) were postulated based on the following:
a. information gathered during plant walkdowns (i.e., fixed or transient ignition source and Thermo-Lag target location, type and quantity of combustibles present in zone);
b. information obtained from reviewing applicable sections in Appendix R documents for Oyster Creek Unit (Ref. 4);
c. information obtained from reviewing Oyster Creek procedure 120.5, Rev. 5, "Control of Combustibles" (Ref. 6)
d. information provided by GPUN engineers (specializing in fire protection, risk assessment, mechanical).

Thermo-Lag FIVE analyses were performed using computerized Excel spreadsheets which duplicate FIVE Worksheets 1,2 , or 3 and are linked to tables containing the same information as in FIVE Reference Tables. For each Thermo-Lag fire modeling scenario, results are presented as exposure time-temperature profiles. In all evaluated scenarios, time-dependent temperatures were derived by incrementally adding the total heat content of the fuel into the fire compartment hot gas layer.

## 3. FIRE MODELING ASSUMPTIONS

The following Thermo-Lag fire modeling assumptions were made:

1. Worst-case fixed fire source scenarios were modeled for each Thermo-Lag sub-configuration. The determination of which fixed fire source might cause the worst-case Thermo-Lag fire scenario was made during the plant fire area walkdown and was based on minimum damage threshold heights and distances calculated for typical fixed fire sources (e.g., electrical cabinets, electrical motors, lube oil in pumps) found at Oyster Creek.
2. Worst-case transient fire source scenarios observed during walkdowns were also modeled. Worst-case transient fire scenarios considered were identical to the types and quantities of transient combustibles identified specifically for TB-FZ-11D in walkdown.
3. To ensure conservative results, worst-case fire plume (or ceiling jet) scenarios and/or radiant flux scenario were first modeled. For T-L subconfigurations not in the plume or ceiling jet, hot gas layer fire scenarios were evaluated.
4. Electrical cable inside cabinets, motor control centers, switchgear and cable trays was assumed to be non-qualified (i.e., non IEEE-383 type). This assumption is conservative since electrical cable purchased and installed at Oyster Creek in the past decade was IEEE-383 type.
5. The Fire Hazard Tool developed for the EPRI Cable Wrap Fire Barrier Tailored Collaboration assumes that the fire compartment temperature will follow the ASTM E-119 time-temperature curve as soon as the compartment hot gas layer (HGL) reaches $1000^{\circ} \mathrm{F}$. When the HGL temperature reaches $1000^{\circ} \mathrm{F}$, the plume (or ceiling jet) temperatures are assumed to follow that of the E-119 (see discussion in Appendix B).
6. The Thermo-Lag wrap is assumed to burn by itself (i.e., even after the initial fire source is out of fuel) unless otherwise stated. Per, the EPRI Method (Ref. 2, Appendix I-D), credit may be taken for a time lag before the Thermo-Lag ignites based on exposure temperature.
7. The maximum plume and ceiling jet temperature is assumed to be $1600^{\circ} \mathrm{F}$, which corresponds to flame temperature. This assumption is valid until hot gas layer temperature exceeds $1600^{\circ} \mathrm{F}$, at which point the plume temperature is assumed to be equal to hot gas layer temperature.
8. Fire Zone TB-FZ-11D specific data:

- total area of $9,668 \mathrm{ft}^{2}$ (Ref. 4 - Oyster Creek Fire Hazard Analysis Report, Rev. No. 5);
- fire zone ceiling height of 20 feet (General Arrangement dwg.);
- ambient temperature of $100^{\circ} \mathrm{F}$.


## 4. FIRE MODELING RESULTS

### 4.1 Thermo-Lag Configuration 1. Scenario 1

The ignition source / combustible considered in this scenario is a Protective Clothing (PC) storage bin (a transient combustible) observed during the plant walkdown. It is assumed that the PC bin ( 3 ft tall) contains 50 lb . of PC's and the plastic container itself, for a total heat content $(Q)$ of 800,000 Btu (from Ref. 5, page E2-1). The heat content assumes 400,000 Btus for the plastic container and 400,000 Btus ( $8000 \mathrm{Btu} / \mathrm{lb}$. ${ }^{*} 50 \mathrm{lb}$.) for the PCs. A heat release rate of $285 \mathrm{Btu} / \mathrm{s}$ was chosen for this fire source based on test data (Ref. 2 App. I-E NBS-Lee, clothing). The test report indicates that there were 10 lb . of clothing used for this test and that the heat release rate was $57 \mathrm{Btu} / \mathrm{s}$. The heat release rate is assumed to increase in proportion to the mass of the PCs. This increase is conservative, because heat release rates are normally more dependent on surface area than on combustible mass. At this heat release rate, the PC bin source will burn for 47 minutes $(800,000 \mathrm{Btu} / 285 \mathrm{Btu} / \mathrm{s})$. The targets in this scenario are two Thermo-Lag wrapped conduits, CGCTB019 and CGPTB020, each 1.5 inch in diameter, (Ref. 1). The Thermo-Lag wrap is assumed to be 0.5 inch thick, with a 6 inch combined total diameter. The targets are located 5 feet above the room floor and 3.5 feet offset from the source/combustible (i.e., in the hot gas layer). However, one T-L wrapped support is located directly in the plume of the fire and is assumed to ignite and burn for the entire scenario. The heat release rate for the T-L wrap is $8.8 \mathrm{Btu} / \mathrm{s} / \mathrm{ft}^{2}$ (Ref. 2, Appendix I-D). The T-L heat release rate of $41.5 \mathrm{Btu} / \mathrm{s}$ was added to that of PC bin fire source ( 285 $\mathrm{Btu} / \mathrm{s}$ ) for the hot gas layer calculations. The virtual surface of the fire is 3 feet above floor level. The fire source is not against the room wall, so the location factor is one.

## Results

The time temperature profile scenario calculated by the model are presented in Table 1. The time-temperature profile shows that the temperature at the target will be at flame temperature for the entire scenario. The target in this case is the Thermo-Lag wrapped support, not the protected conduits themselves. The conduit exposure is expected to be only hot gas layer because it is outside of the plume region. The model results indicate that hot gas layer effects in the zone due to this postulated fire are negligible. A sample model worksheet is provided as Appendix A, Figure A-1. The time-temperature profile for this scenario is shown in Table 1.

Table 1. Temperature-Time History for Fire Scenario Modeled in Configuration 1, Scenario 1.

| Time | HRR to <br> Targot | HRR to HGL | QTOT | Temperature | Temperature |
| :---: | :---: | :---: | :---: | :---: | :---: |
| (min.) | (Btu/s) | (Btu/s) | (Btu) | Target -(F) | HGL-(F) |
| 0 | 285 | 327 | 0 | 1600 | 100 |
| 5 | 285 | 327 | 98100 | 1600 | 102 |
| 10 | 285 | 327 | 196200 | 1600 | 104 |
| 15 | 285 | 327 | 294300 | 1600 | 106 |
| 20 | 285 | 327 | 392400 | 1600 | 108 |
| 25 | 285 | 327 | 490500 | 1600 | 111 |
| 30 | 285 | 327 | 588600 | 1600 | 113 |
| 40 | 285 | 327 | 784800 | 1600 | 117 |
| 47 | 285 | 327 | 922140 | 1600 | 120 |
| 48 | 0 | 42 | 924680 | 1600 | 120 |
| 60 | 0 | 42 | 954900 | 1600 | 121 |

### 4.2 Thermo-Lag Configuration 1. Scenario 2

Scenario two postulates a fire in a $3^{\prime}$ wide electrical cabinet on the wall, 4 feet off the ground. The location factor for the fire is two. The target for the scenario is an unwrapped $12^{\prime \prime}$ cable tray 7 feet directly above the panel, 11 feet off the ground. A second $24^{\prime \prime}$ cable tray is located 1 foot above the first cable tray. In addition to the vertical propagation, the fire is also assumed to propagate horizontally along the exposed cable tray with a flame spread rate of 10 feet per hour. The T-L of concern for this scenario is configuration 1 which are 6 feet offset from the cable trays and three feet below the ceiling. They will see only hot gas layer exposure.

The heat release rates used for this scenario are as follows:

1. electrical panel with non qualified cable $-400 \mathrm{Btu} / \mathrm{s}$ (Ref. 3); this HRR is applicable to scenario time $t=0$ to 15 minutes;
2. first cable tray $-23.4 \mathrm{Btu} / \mathrm{s}-\mathrm{ft}$. ${ }^{2}$ (Ref. 2, FIVE Table 1 E , cable \# 5 adjusted to consider Equation 2 on FIVE manual, page $10.4-10$ ); since $3 \mathrm{ft.}^{2}$ ( 3 linear feet, 1 foot wide) are initially burning, the HRR for this source alone is $70 \mathrm{Btu} / \mathrm{s}$. An additional 23.4 $\mathrm{Btu} / \mathrm{s}$ are added every 6 minutes to account for fire propagation;
3. second cable tray $-23.4 \mathrm{Btu} / \mathrm{s}-\mathrm{ft}$. ${ }^{2}$ (Ref. 2, FIVE Table 1 E , cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); $10 \mathrm{ft}^{2}$ ( 5 linear feet, 2 feet wide) are burning, therefore the HRR for this second cable tray is $234 \mathrm{Btu} / \mathrm{s}$, and additional $47 \mathrm{Btu} / \mathrm{s}$ are added every 6 minutes to account for fire propagation.

The total available heat $\left(Q_{\text {von }}\right)$ from each fire source is:

1. electrical panel $-360,000$ Btus (assumes that the panel contains enough fuel to sustain a 15 minute fire);
2. first cable tray $-2,400,000$ Btus [assumes $10,000 \mathrm{Btu} / \mathrm{lb}$. of cable tray (per Ref. 5, page E2-1) $\times 10$ linear feet $\times 24 \mathrm{lb}$./linear foot (engineering judgment for $100 \%$ filled, 12 inch wide cable tray)]. The fuel will last for the full one hour duration of the fire;
3. second cable tray $-4,000,000$ Btus [assumes $10,000 \mathrm{Btu} / \mathrm{lb}$. of cable tray (per Ref. 5, page E2-1) $\times 10$ linear feet $\times 40 \mathrm{lb}$. / linear foot (engineering judgment for $100 \%$ filled, 24 inch wide cable tray)]. The fuel will last for the full one hour duration of the fire.

## Results

The results are presented in Table 2. The model results indicate that hot gas layer effects in the zone due to this postulated fire cause the temperature to rise to approximately $322^{\circ} \mathrm{F}$. A sample model worksheet is provided in Appendix A, Figure A-2.

Table 2. Temperature-Time History for Fire Scenario Modeled in Configuration 1, Scenario 2.

| Time | HRR to <br> Target | HRR to HGL | QTOT | Temperature | Temperature |
| :---: | :---: | :---: | :---: | :---: | :---: |
| (min.) | (Btu/s) | $($ Btu/s) | (Btu) | Target - (F) | HGL - (F) |
| 0 | 704 | 704 | 0 | 100 | 100 |
| 5 | 774 | 774 | 232260 | 111 | 111 |
| 10 | 844 | 844 | 485580 | 123 | 123 |
| 15 | 915 | 915 | 759960 | 136 | 136 |
| 16 | 985 | 985 | 819048 | 138 | 138 |
| 25 | 1055 | 1055 | 1388748 | 167 | 167 |
| 30 | 1125 | 1125 | 1726308 | 184 | 184 |
| 35 | 1195 | 1195 | 2084928 | 203 | 203 |
| 40 | 1266 | 1266 | 2464608 | 224 | 224 |
| 50 | 1336 | 1336 | 3266088 | 270 | 270 |
| 60 | 1406 | 1406 | 4109688 | 322 | 322 |

### 4.3 Thermo-Lag Configuration. 1. Scenario 3

This fire scenario modeled a fire in an oil-filled transformer near 460 V unit sub-station 1-A-1. The transformer contains 190 gallons of mineral oil (from the plant walkdown). Each gallon of oil is equivalent to 125,656 Btus (from Ref. 3, Table 2E), for a total of about 23.9 million Btus for the entire volume of oil in the transformer. The heat release rate for the transformer oil is $135 \mathrm{Btu} / \mathrm{s}$-ft ${ }^{2}$ (from Ref. 2, Table 2E). The oil could spread outside the transformer over an area of $98.5 \mathrm{ft}^{2}$ (measured during the plant walkdown). The heat release rate assumed for this fire scenario is $13,300 \mathrm{Btus} / \mathrm{s}$ ( $135 \mathrm{Btu} / \mathrm{s}-\mathrm{ft}^{2} \times 98.5$ $\mathrm{ft}^{2}$ ). The fire duration is then ( $23,9 \mathrm{M} \mathrm{Btu} / 13,300 \mathrm{But} / \mathrm{s}$ ) 30 minutes. The nearest T-L target is 1 foot off the room ceiling and 15 feet offset from the edge of the postulated
target is 1 foot off the room ceiling and 15 feet offset from the edge of the postulated transformer oil pool fire. The T-L target in this scenario is configuration 1. It consists of two conduits, CGCTB019 and CGPTB020, each 1.5 inch in diameter, wrapped in Thermo-Lag (Ref. 1), with a 6 inch combined total diameter. They are in the ceiling jet region of the fire. When the exposure temperature at the targets reaches $1000^{\circ} \mathrm{F}$, the Thermo-Lag is assumed to ignite. A twenty foot length of Thermo-Lag with a surface area of $\left(20^{\prime} * 0.5^{\prime} \pi\right) 31.4 \mathrm{ft}^{2}{ }^{2}$ is assumed to start burning with a heat release rate of (8.8 $\mathrm{Btu} / \mathrm{s} / \mathrm{ft}^{2}$ * $\left.31.4 \mathrm{ft}^{2}{ }^{2}\right) 276 \mathrm{Btu} / \mathrm{s}$, which contributes to the hot gas layer. Because the Thermo-Lag is burning when the oil fire goes out, the exposure at the Thermo-Lag is conservatively assumed to remain the same for the remaining fire duration.

## Results

The time temperature profile for this model shows that the temperature at the target will reach approximately $1278^{\circ} \mathrm{F}$ due to a fire of this nature. This temperature is above the ignition temperature of Thermo-Lag. The model results indicate that hot gas layer effects in the zone due to this postulated fire will not reach flashover in the zone. A sample model worksheet is provided in Appendix A, Figure A-3.

Table 3. Temperature-Time History for Fire Scenario Modeled in Configuration 1 Scenario 3.

| Time | HRR to <br> Target | HRR to HGL | QTOT | Temperature | Temperature |
| :---: | :---: | :---: | :---: | :---: | :---: |
| $($ min. $)$ | $($ Btu/s $)$ | $($ Btu/s $)$ | $($ Btu $)$ | Target $-($ F $)$ | HGL $-(\mathrm{F})$ |
| 0 | 13300 | 13300 | 0 | 613 | 100 |
| 5 | 13300 | 13300 | 3990000 | 690 | 177 |
| 10 | 13300 | 13300 | 7980000 | 778 | 266 |
| 15 | 13300 | 13300 | 11970000 | 879 | 366 |
| 20 | 13300 | 13300 | 15960000 | 993 | 480 |
| 22 | 13300 | 13576 | 17589120 | 1044 | 532 |
| 30 | 13300 | 13576 | 24105600 | 1278 | 765 |
| 31 | 0 | 276 | 24122160 | 1278 | 766 |
| 40 | 0 | 276 | 24271200 | 1278 | 772 |
| 50 | 0 | 276 | 24436800 | 1278 | 779 |
| 60 | 0 | 276 | 24602400 | 1278 | 785 |

### 4.4 Thermo-Lag Configuration 2. Scenario 1

This fire scenario models a fixed ignition source/combustible. The source is a 2 foot wide cable tray with non IEEE- 383 cable, 16 feet off the room's floor. Since the cable is non-qualified, the cable fire can self-ignite (per Ref. 2, Appendix I-A) and start to burn. The fire will then propagate immediately to a second cable tray, located 1 foot above it. In addition to the vertical propagation, the fire is also assumed to propagate horizontally along the exposed cable tray with a flame spread rate of 10 feet per hour (Ref. 2 Appendix I-C). The total amount of cable tray assumed to be available for this
fire scenario is 10 feet for a 1 hour fire duration. The cable trays are only about $50 \%$ filled (visual observation during plant walkdown). The targets are two 2 -inch diameter conduits wrapped in 0.5 inch thick Thermo-Lag. The T-L wrapped conduits pass perpendicularly over the cable trays, 3 feet above them. Due to the close distance between the cable trays and T-L wrapped conduits, the T-L will be ignited by the cable tray fire (assumed conservatively to happen at time $t=0$ minutes).

The heat release rates considered for this scenario are as follows (by fire source):

1. first cable tray $-23.4 \mathrm{Btu} / \mathrm{s}-\mathrm{ft}$. ${ }^{2}$ (Ref. 2, FIVE Table 1 E , cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); since $4 \mathrm{ft}^{2}$ (2 linear ft ) are assumed to be initially burning, the HRR for this source alone is' $94 \mathrm{Btu} / \mathrm{s}$; this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be $140 \mathrm{Btu} / \mathrm{s}$, which includes the HRR from $2 \mathrm{ft}^{2}$ of additional cable tray);
2. second cable tray $-23.4 \mathrm{Btu} / \mathrm{s}-\mathrm{ft}$. ${ }^{2}$ (Ref. 2, FIVE Table 1 E , cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); the HRR for this source alone is $187 \mathrm{Btu} / \mathrm{s}$ since $8 \mathrm{ft}^{2}$ (4 linear ft .) are assumed initially burning; this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be $234 \mathrm{Btu} / \mathrm{s}$, which includes the HRR from 2 $\mathrm{ft}^{2}$ of additional cable tray);
3. T-L wrap HRR is assumed to be $8.8 \mathrm{Btu} / \mathrm{s}-\mathrm{ft} .^{2}$ (per Ref. 2 Appendix I-D); the amount of Thermo-Lag that could be involved in this fire scenario is assumed to be 10 linear feet (visual observation) with a circumference of $\left(2^{*} .25^{\prime *} \pi\right) 1.6 \mathrm{ft}$. for a total surface area of $16 \mathrm{ft}^{2}$; the HRR is thus $141 \mathrm{Btu} / \mathrm{s}$; this HRR will be considered only for calculating the HGL temperature rise.

The total available heat $\left(\mathrm{Q}_{100}\right)$ from each fire source is:

1. first cable tray - $2,000,000$ Btus [assumes $10,000 \mathrm{Btu} / \mathrm{lb}$. of cable tray (per Ref. 5 , page E2-1) $\times 10$ linear feet $\times 20 \mathrm{lb}$. /linear foot (engineering judgment for $50 \%$ filled, 24 inch wide cable tray)]; This source has enough combustible material to burn for one hour.
2. second cable tray $-2,000,000$ Btus [assumes $10,000 \mathrm{Btu} / \mathrm{lb}$. of cable tray (per Ref. 5, page E2-1) $\times 10$ linear feet $\times 20 \mathrm{lb}$./linear foot (engineering judgment for $50 \%$ filled, 24 inch wide cable tray)]; This source has enough combustible material to burn for one hour.
3. Thermo-Lag wrap - 588,000 Btus (assumes 58,800 Btu/linear foot, from Ref. 7).

## Results

The time temperature profile shows that the temperature at the target reaches flame temperature immediately, and remains at that temperature for the remainder of the fire (1 hour). This iemperature is above the ignition temperature of Thermo-Lag. The
model results indicate that hot gas layer temperature will reach $702^{\circ} \mathrm{F}$ after an hour. A sample model worksheet is provided in Appendix A, Figure A-4.

Table 4. Temperature-Time History for Fire Scenario Modeled in Configuration 2, Scenario 1.

| Time | HRR to <br> Target | HRR to HGL | QTOT | Temperature | Temperature |
| :---: | :---: | :---: | :---: | :---: | :---: |
| $(\mathrm{min})$ | $(\mathrm{BTU} / \mathrm{s})$ | $(\mathrm{BTU} / \mathrm{s})$ | $($ BTU $)$ | Target - (F) | HGL - (F) |
| 0 | 281 | 422 | 0 | 1600 | 100 |
| 6 | 375 | 516 | 185616 | 1600 | 123 |
| 12 | 468 | 609 | 404928 | 1600 | 151 |
| 18 | 562 | 703 | 657936 | 1600 | 186 |
| 24 | 655 | 796 | 944640 | 1500 | 227 |
| 30 | 749 | 890 | 1265040 | 1600 | 276 |
| 36 | 843 | 984 | 1619136 | 1600 | 335 |
| 42 | 936 | 1077 | 2006928 | 1600 | 405 |
| 48 | 1030 | 1171 | 2428416 | 1600 | 487 |
| 54 | 1123 | 1264 | 2883600 | 1600 | 586 |
| 60 | 1217 | 1358 | 3372480 | 1600 | 702 |

### 4.5 Thermo-Lag Configuration 2 Scenario 2

This fire scenario fixed ignition source/combustible is a 2 foot wide electrical cabinet with exposed vertical cable igniting a 24 inch wide cable tray, 14 feet above the room's floor. The scenario postulates that the fire propagates horizontally along the cable tray at a rate of 10 feet per hour (since cable is not IEEE-383 qualified), but away from the nearest Thermo-Lag targets. The nearest T-L targets are located approximately 5 feet offset from the cable tray and at about the same height (i.e., 14 feet off floor). They are below the ceiling jet and will experience hot gas layer exposure only. The Thermo-Lag targets considered in this scenario are configuration 2.

The heat release rates considered for this scenario are as follows (by fire source):

1. electrical panel with non qualified cable $-400 \mathrm{Btu} / \mathrm{s}$ (Ref. 3); this HRR is applicable to scenario time $\mathrm{t}=0-15$ minutes;
2. cable tray - $23.4 \mathrm{Btu} / \mathrm{s}-\mathrm{ft}^{2}$ (Ref. 2, FIVE Table 1E, cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); since $4 \mathrm{ft}^{2}$ are initially burning, the HRR for this source alone is $94 \mathrm{Btu} / \mathrm{s}$; this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be $140 \mathrm{Btu} / \mathrm{s}$, which includes the HRR from $2 \mathrm{ft}^{2}$ of additional cable tray).

The total available heat $\left(Q_{\text {wot }}\right)$ from each of the two fire sources is:

1. electrical panel - 360,000 Btus (it assumes that the panel contains enough fuel to sustain a 15 minute fire);
2. cable tray $-2,000,000$ Btus [it assumes $10,000 \mathrm{Btu} / \mathrm{lb}$. of cable tray (per Ref. 5, page E2-1) $\times 10$ linear feet $\times 20 \mathrm{lb}$. /linear foot (engineering judgment for $50 \%$ filled, 24 inch wide cable tray)]. This is enough combustible material for one hour.

## Results

The results are presented in Table 5. The time temperature history shows that the temperature at the target will not increase appreciably, reaching only $202^{\circ} \mathrm{F}$. A sample model worksheet is provided as Appendix A, Figure A-5.

Table 5. Temperature-Time History Fire Scenario Modeled in Configuration 2, Scenario 2.

| Time | HRR to <br> Target | HRR to HGL | QTOT | Temperature | Temperature |
| :---: | :---: | :---: | :---: | :---: | :---: |
| $($ min. $)$ | $($ Btu/s) | $($ Btw/s $)$ | $($ Btu $)$ | Target - $(F)$ | HGL - $(\mathrm{F})$ |
| 0 | 494 | 494 | 0 | 100 | 100 |
| 6 | 540 | 540 | 194544 | 112 | 112 |
| 12 | 587 | 587 | 405936 | 125 | 125 |
| 18 | 234 | 234 | 490176 | 131 | 131 |
| 24 | 281 | 281 | 591264 | 137 | 137 |
| 30 | 328 | 328 | 709200 | 145 | 145 |
| 36 | 374 | 374 | 843984 | 154 | 154 |
| 42 | 421 | 421 | 995616 | 164 | 164 |
| 48 | 468 | 468 | 1154096 | 175 | 175 |
| 54 | 515 | 515 | 1349424 | 188 | 188 |
| 60 | 562 | 562 | 1551600 | 202 | 202 |

### 4.6 Thermo-Lag Configuration 2 Scenario 3

This fire scenario modeled a fixed ignition souice,' combustible, a 6 inch wide cable tray with non IEEE-383 cable, 18 feet off the room's floor. Since the cable is non-qualified, the cable fire can self-ignite (per Ref. 2, Appendix I-A) and start to burn. The fire will then propagate horizontally along the exposed cable tray with a flame spread rate of 10 feet per hour (per Ref. 11). The total amount of cable tray assumed to be available for this fire scenario is 10 feet (for a 1 hour fire duration). The cable tray is only about $25 \%$ filled (visual observation during plant walkdown). The targets are two 2 -inch diameter conduits wrapped in 0.5 inch thick Thermo-Lag. The T-L wrapped conduits are routed in the same direction as the cable tray, 8 inches above it. Due to the close distance between the cable tray and T-L wrapped conduits, the T-L will be ignited by the cable tray fire (assumed conservatively to happen at time $t=0$ minutes). Once this happens, additional heat is released into this Turbine Building Basement fire zone.

The heat release rates considered for this scenario are as follows (by fire source):

1. cable tray $-23.4 \mathrm{Btu} / \mathrm{s}-\mathrm{ft}^{2}$ (Ref. 3, FIVE Table 1E, cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10). Two iinear feet of the $6^{\prime \prime}$ wide tray is assumed to burn, so a heat release rate of $23 \mathrm{Btu} / \mathrm{s}$ is used. This HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be $35 \mathrm{Btu} / \mathrm{s}$, which includes the HRR from $0.5 \mathrm{ft}^{2}$ of additional cable tray);
2. T-L wrap HRR is $8.8 \mathrm{Btu} / \mathrm{s}$-ft ${ }^{2}$ (per Ref. 2, Appendix I-D). The amount of Thermo-Lag assumed to burn is that portion of the Thermo-Lag directly above the burning cable tray. The circumference of the Thermo-Lag wrapped conduit is 1.6 feet, hence, the HRR for the Thermo-Lag is 1.6 ft . " 1 ft . " $8.8 \mathrm{Btu} / \mathrm{s} / \mathrm{ft} .2=$ 14 Btu/linear foot. Two feet will be burning initially plus an additional foot every 6 minutes. This HRR will be considered only for calculating the HGL temperature rise

The total available heat $\left(Q_{\text {net }}\right)$ from each fire source is:

1. first cable tray - 270,000 Btus [assumes $10,000 \mathrm{Btu} / \mathrm{lb}$. of cable tray (per Ref. 5, page E2-1) $\times 12$ linear feet $\times 3 \mathrm{lb}$. /linear foot (engineering judgment for $25 \%$ filled, 6 -inch wide cable tray) * $70 \%$ combustion];
2. Thermo-Lag wrap $-706,000$ Btus (assumes $58,800 \mathrm{Btu} /$ linear foot * 12 feet, from Ref. 7).

## Results

Table 6. Temperature-Time History Fire Scenario Modeled in Configuration 2, Scenario 3.

| Time | HRR to <br> Target | HRR to HGL | QTOT | Temperature | Temperature |
| :---: | :---: | :---: | :---: | :---: | :---: |
| $(\mathrm{min})$ | $(\mathrm{BTU} / \mathrm{s})$ | $($ BTU/s) | $($ BTU $)$ | Target - (F) | HGL - (F) |
| 0 | 23 | 52 | 0 | 1600 | 100 |
| 5 | 35 | 77 | 23202 | 1600 | 104 |
| 10 | 47 | 103 | 54138 | 1600 | 190 |
| 15 | 59 | 129 | 92808 | 1600 | 117 |
| 16 | 70 | 155 | 102089 | 1600 | 119 |
| 25 | 82 | 180 | 199537 | 1600 | 137 |
| 30 | 94 | 206 | 261409 | 1600 | 150 |
| 35 | 105 | 232 | 331015 | 1600 | 164 |
| 40 | 117 | 258 | 408355 | 1600 | 179 |
| 50 | 129 | 284 | 578503 | 1600 | 216 |
| 60 | 140 | 309 | 764119 | 1600 | 258 |

The results are presented in Table 6. The T-L wrapped target is at flame temperature for the entire scenario duration. The hot gas layer will reach approximately $234^{\circ} \mathrm{F}$. A sample model worksheet is provided as Appendix A, Figure A-6.

### 4.7 Thermo-Lag Configuration 2.Scenario 4

This fire scenario modeled a transient ignition source/combustible, a 5 gallon container containing lube oil. Per Oyster Creek Procedure "Control of Combustibles," Procedure No. 120.5, Rev. 4 (Ref. 6), combustible liquids could be transported throughout the plant in an approved closed metal container (but not necessarily a safety-can). For this fire scenario, it was postulated that 1 quart of lube oil is spilled out of a can on the floor and is not cleaned up. The oil is later ignited by a welder working nearby or another transient ignition source and begins to burn. The oil fire could potentially impact a T-L wrap 14. feet above the ground.

The heat release rate for the lube oil is $110 \mathrm{Btu} / \mathrm{s}-\mathrm{ft}^{2}$ (from Ref. 3, Table 2E). The oil could spread unconfined over an area of $13.5 \mathrm{ft}^{2}$ ( $54 \mathrm{ft}^{2} / \mathrm{gal}$ from Ref. 3, Table 3E). The heat release rate assumed for this fire scenario is $1,485 \mathrm{Btu} / \mathrm{s}\left(110 \mathrm{Btu} / \mathrm{s}-\mathrm{ft}^{2} \times 13.5 \mathrm{ft}^{2}\right)$. The heat content of one quart of oil is ( $17,111 \mathrm{Btu} / \mathrm{lb} . * 60 \mathrm{lb} . / 1 \mathrm{ft}^{3} .{ }^{*} 1 \mathrm{ft}^{3} / 30$ quarts $)$ 34,222 Btu (from Ref. 3, Table 2E). The oil fire duration will be less than one minute, but one minute will be used for conservatism. The virtual surface of the fire is at the floor level. The location factor for this source is one.

## Results

The short oil fire does not cause an exposure temperature high enough to ignite the Thermo-Lag. The fire is of a short duration, with almost no hot gas layer effects. A sample model worksheet is provided as Appendix A, Figure A-7.

Table 7. Temperature-Time History for Fire Scenario Modeled in Configuration 2, Scenario 4.

| Tirne | HRR to <br> Target | HRR to HGL | QTOT | Temperature | Temperature |
| :---: | :---: | :---: | :---: | :---: | :---: |
| $($ min. $)$ | $($ Btu/s) | $($ Btu/s $)$ | $($ Btu $)$ | Target - (F) | HGL - (F) |
| 0 | 1485 | 1485 | 0 | 964 | 100 |
| 1 | 1485 | 1485 | 89100 | 965 | 102 |
| 2 | 0 | 0 | 89100 | 102 | 102 |
| 10 | 0 | 0 | 89100 | 102 | 102 |
| 15 | 0 | 0 | 89100 | 102 | 102 |
| 25 | 0 | 0 | 89100 | 102 | 102 |
| 30 | 0 | 0 | 89100 | 102 | 102 |
| 35 | 0 | 0 | 89100 | 102 | 102 |
| 40 | 0 | 0 | 89100 | 102 | 102 |
| 50 | 0 | 0 | 89100 | 102 | 102 |
| 60 | 0 | 0 | 89100 | 102 | 102 |

## 5. FIRE MODELING SUMMARY

Seven scenarios were modeled for the two T-L configurations in Oyster Creek Fire Zone TB-FZ-11D.. The analysis results are presented as time- temperature profiles. These results are used in the following section to develop total heat load and then convert to equivalent ASTM E-119 exposures.

## 6. APPROXIMATING EXPOSURE TIME-HISTORY OF THERMO-LAG SUB-CONFIGURATIONS -- THE TOTAL HEAT LOAD CONCEPT

Calculation of an exposure time-temperature profile for a specific T-L Subconfiguration is based on a the methodology presented in Reference 2. The time temperature profiles are used to develop the total heat load (Reference 2, Appendix IIIA) for each fire exposure. The total heat load concept holds that the area under the incident heat flux vs. time curve, or total heat load, can be used as a measure of fire severity. Comparing the total heat load of a fire scenario exposure to the total heat load of a test exposure (ASTM E-119) allows one to predict the response of a Thermo-Lag barrier to the scenario exposure. The total heat load per unit area of fire barrier surface is equal to the area under the incident heat flux-time curve, corrected for convection effects as described in Reference 2, Appendix III-A.

### 6.1 Total Heat Load Results

| Raceway <br> configurat <br> ion | Scenario | Description | Peak Temperature <br> at Target ( $\left.{ }^{\circ} \mathrm{F}\right)$ | Total Heat Load <br> Equivalent E-119 <br> Exposure Duration <br> (min.) |
| :--- | :--- | :--- | :--- | :--- |
| 1 | 1 | Protective Clothing | 1600 | $>60$ |
| 1 | 2 | Electrical Cabinet and <br> Cable Tray | 322 | 10 |
| 1 | 3 | Transformer | 1278 | 40 |
| 2 | 1 | Cable Tray | 1600 | $>60$ |
| 2 | 2 | Electrical Cabinet and <br> Cable Tray | 202 | 10 |
| 2 | 3 | Cable Tray | 1600 | $>60$ |
| 2 | 4 | Oil transient | 965 | 10 |

Appendix C contains tables with Total Heat Load values calculated for Thermo-Lag sub-configurations 1 , and 2.

## 7. REFERENCES

1. Oyster Creek Plant Drawing of Turbine Building Basement.
2. EPRI Report "Methods for Evaluation of Cable Wrap Fire Barrier Performance", July, 1995
3. EPRI TR-100370, "Fire-Induced Vulnerability Evaluation," April 1992.
4. GPUN, "Fire Hazard Analysis Report - Oyster Creek," Doc. Non. 990-1746 Rev. 8.
5. GPUN - TMI-1 Procedure No. 1035 (Revision 24), August 23, 1994.
6. GPUN- Oyster Creek Procedure No. 120.5, Rev. 5, "Control Of Combustibles," May 10, 1993.
7. Thermo-Lag 330-1 Combustibility Study, NEI, 1994.
8. "Methods for Evaluating Cable Wrap Fire Barrier Performance," EPRI draft report developed under the Tailored Collaboration Project, October 1994.
9. EPRI TR-100443, "Methods of Quantitative Fire Hazard Analysis," EPRI final report, May 1992.
10. EPRI NP-7332, "Design Guide for Fire Protection of Grouped Electrical Cables," EPRI Final Report, May 1991.

## Appendix A

## Sample Scenario Worksheets

Figure A-1

| Temperature calculation at time |  | min | 5 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | ft2 | 9668 |
| 1 | Location of Target |  | Plume |
| 2 | Height of Target above Fire Source | ft | 2.00 |
| 3 | Height from Fire Source to Ceiling, H | ft | 17.00 |
| 4 | Ratio of Target Height/Ceiling Height |  | 0.118 |
| 5 | Longitudinal Distance from Fire Source to Target. L | ft | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | $f$ | NA |
| 8 | Height to Width Ratio, HW |  | NA |
| 9 | Peak Fire Intensity | Btu/s | 285 |
| 10 | Fire Location Factor |  | 1 |
| 11 | Effective Fire Intensity | Btw/s | 285 |
| 12 | Plurne Temperature Rise at Target | F | 1600 |
| 13 | Temp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | F | 1600 |
| 15 | Temperature at Target | F | 1600 |
| 16 | Temperature in HGL | F | 102 |
| 17 | Otot to HGL | Btu | 98100 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated Onet | Btu | 5886 |
| 20 | Calculated Enclosure Volume, V | H3 | 164356 |
| 21 | Calculated QnetV | Qnetht | 0.04 |
| 22 | HGL. Temperature Increase | F | 2 |
|  |  |  |  |

Figure A-2

| Temperature calculation at time |  | min | 5 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | f12 | 9668 |
| 1 | Location of Target |  | Hiot Gas Layer |
| 2 | Height of Target above Fire Source | f | 5.00 |
| 3 | Height from Fire Source to Ceiling, H | f | 8.00 |
| 4 | Ratio of Target Height/Ceiling Height |  | 0.625 |
| 5 | Longitudinal Distance from Fire Source to Target, L | H | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclos ure Width, W | f | NA |
| 8 | Height o Width Ratio, HWW |  | NA |
| 9 | Peak Fire Intensity | Btws | 774 |
| 10 | Fire Location Factor |  | 2 |
| 11 | Effective Fire Intensity | Btw/s | 1548 |
| 12 | Plume T emperature Rise at Target | F | 1600 |
| 13 | Temp Rise Factor at Target |  | 0.00 |
| 14 | Temperature Rise at Target | F | 0 |
| 15 | Temperalure at Target | F | 111 |
| 16 | Temperature in HGL | F | 111 |
| 17 | Qtot to HGL | Btu | 232260 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated 2 net | Btu | 13936 |
| 20 | Calculated İnclosure Volume, V | $\mathrm{ft3}$ | 77344 |
| 21 | Calculated (2netV | Qnetft3 | 0.18 |
| 22 | HGL Tempe ature increase | F | 11 |
|  |  |  |  |

Figure A-3

| Temperature calculation at time |  | min | 5 |
| :---: | :---: | :---: | :---: |
| Maxirnum ambient ternperature |  | F | 100 |
| Floor Area |  | H2 | 9668 |
| 1 | Location of Target |  | Ceiling Jet |
| 2 | Height of Target above Fire Source | H | 19.00 |
| 3 | Height from Fire Source to Ceiling, H | H | 20.00 |
| 4 | Ratio of Target Heigh/Ceiling Height |  | 0.950 |
| 5 | Longitudinal Distance from Fire Source to Target L | H | 15 |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | 0.75 |
| 7 | Enclosure Width, W | H | 112 |
| 8 | Height to Width Ratio, H/W |  | 0.18 |
| 9 | Peak Fire Intensity | Btu/s | 13300 |
| 10 | Fire Location Factor |  | 1 |
| 11 | Effective Fire Intensity | Btw/s | 13300 |
| 12 | F.ime Temperature Rise at Target | F | 1411 |
| 13 | Ternp Rise Factor at Target |  | 0.36 |
| 14 | Temperature Rise at Target | F | 513 |
| 15 | Temperature at Target | F | 690 |
| 16 | Temperature in HGL. | F | 177 |
| 17 | Qtot to HGL | Btu | 3990000 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated Qnet | Btu | 239400 |
| 20 | Calculated Enclosure Volume, V | H3 | 193360 |
| 21 | Calculated OnetV | Qnet/t3 | 1.24 |
| 22 | HGL Temperature Increase | F | 77 |
|  |  |  |  |

Figure A-4
SAIC Report No. 01-0082-05-2965-700-1

| Temperature calculation at time |  | min | 6 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | H2 | 9668 |
| 1 | Location of Target |  | Plume |
| 2 | Height of Target above Fire Source | 1 | 2.90 |
| 3 | Height from Fire Source to Ceiling, H | ${ }^{\prime \prime}$ | 3.00 |
| 4 | Ratio of Target Heigh/Ceiling Height |  | 0.967 |
| 5 | Longtudinal Distance from Fire Source to Target. 1 | $f$ | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H. |  | NA |
| 7 | Enclosure Width, W | H | NA |
| 8 | Height to Width Ratio, H/W |  | NA |
| 9 | Peak Fire Intensity | Btw/s | 375 |
| 10 | Fire Location Factor |  | 1 |
| 11 | Efiective Fire Intensity | Btu/s | 375 |
| 12 | Plume Temperature Rise at Target | F | 1600 |
| 13 | Temp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | F | 1600 |
| 15 | Temperature at Target | F | 1600 |
| 16 | Temperature in HGL | $F$ | 123 |
| 17 | Otot to HGL | Btu | 185616 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated Qnet | Btu | 11137 |
| 20 | Calculated Enclosure Volume, V | H3 | 29004 |
| 21 | Calculated OnetV | Onet/t3 | 0.38 |
| 22 | HGL Temperature increase | F | 23 |
|  |  |  |  |

Figure A-5
SAIC REport No. 01-0082-05-2965-700-1

| Temperature calculation at time |  | min | 6 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | H2 | 9668 |
| 1 | Location of Target |  | Hot Gas Layer |
| 2 | Height of Target above Fire Source | ft | 0.00 |
| 3 | Height from Fire Source to Ceiling, H | 1 | 6.00 |
| 4 | Ratio of Target Height/Ceiling Height |  | 0.000 |
| 5 | Longitudinal Distance from Fire Source to Target. 1 | H | NA |
| 6 | Longitudinal Distance to Height Ratio, LHH |  | NA |
| 7 | Enclosure Width, W | $f$ | NA |
| 8 | Height to Width Ratio, HWW |  | NA |
| 9 | Peak Fire Intensity | Btu/s | 540 |
| 10 | Fire Location Factor |  | 1 |
| 11 | Effective Fire Intensity | Btws | 540 |
| 12 | Plume Temperature Rise at Target | F | 0 |
| 13 | Temp Rise Factor at Target |  | 0.00 |
| 14 | Temperature Rise at Target | F | 0 |
| 15 | Temperature at Target | $F$ | 112 |
| 16 | Temperature in HGL. | F | 112 |
| 17 | Qtot to HGL. | Btu | 194544 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Caiculated Onet | Btu | 11673 |
| 20 | Caiculated Enclosure Volume, V | $f$ f3 | 58008 |
| 21 | Calcuiated QnetV | Onet/ti3 | 0.20 |
| 22 | HGL. Temperature increase | F | 12 |
|  |  |  |  |


| Temperature Calculation at time |  | min | 5 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | $f 2$ | 9668 |
| 1 | Location of Target |  | Plume |
| 2 | Height of Target above Fire Source | H | 0.66 |
| 3 | Height from Fire Source to Ceiling, H | f1 | 2.00 |
| 4 | Rasio of Target Height/Ceiling Height |  | 0.330 |
| 5 | Longitudinal Distance from Fire Source to Taraet. L | $\pi$ | NA |
| 6 | ILongitudinal Distance to Height Ratio, LHH . |  | NA |
| 7 | Enciosure Widin, W | H | NA |
| 8 | Height to Width Ratio, H/W |  | NA |
| 9 | Peak Fire Intensity | Btu/s | 35 |
| 10 | Fire Location Factor |  | 1 |
| 11 | Effective Fire Intensity | Btu/s | 35 |
| 12 | Plume Temperature Riss at Target | F | 1600 |
| 13 | Temp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | F | 1600 |
| 15 | Temperature at Target | F | 1600 |
| 16 | Temperature in HGL | F | 104 |
| 17 | Otot to HGL | Btu | 23202 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated Onet | Btu | 1392 |
| 20 | Calculated Enclosure Volume, V | H3 | 19336 |
| 21 | Calculated Onet/V | Onet/ti3 | 0.07 |
|  | HGL Ternperature Increase | F | 4 |
|  |  |  |  |

Figure A-7

| Temperature Caiculation at time |  | min | 1 |
| :---: | :---: | :---: | :---: |
| Maximum ambient termperature |  | F | 100 |
| Foor Area |  | H2 | 9668 |
| 1 | Location of Target |  | Plume |
| 2 | Height of Target above Fire Source | $f$ | 14.00 |
| 3 | Height from Fire Source to Ceiling, H | ft | 20.00 |
| 4 | Ratio of Target Height/Ceiling Height |  | 0.700 |
| 5 | Longitudinal Distance from Fire Source to Target. 1 | ft | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | H | NA |
| 8 | Height to Width Ratio, H/W |  | NA |
| 9 | Peak Fire Intensity | Btw/s | 1485 |
| 10 | Fire Location Factor |  | 2 |
| 11 | Effective Fire Intensity | Btw/s | 2970 |
| 12 | Plume Temperature Rise at Target | F | 864 |
| 13 | Temp Rise Factor at Target |  | 1.00 |
| 14 | Temperature Rise at Target | F | 864 |
| 15 | Temperature at Target | F | 965 |
| 16 | Temperature in HGL | F | 102 |
| 17 | Otot to HGL | Btu | 89100 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Caiculated Onet | Btu | 5346 |
| 20 | Caiculated Enclosure Volume, V | $\mathrm{H}_{3}$ | 193360 |
| 21 | Caiculated Qnet/V | Qnet/13 | 0.03 |
| 22 | HGL Temperature increase | F | 2 |
|  |  |  |  |

## Appendix B

Basis for Following ASTM E-119 Time-Temperature Relationship When T(HGL) is $1000^{\circ} \mathrm{F}$

## TRANSITIION TO

## ASTM E-119 TIME-HISTORY IN THE HOT GAS LAYER

## ISSUE:

The Fire Hazard Tool currently under development in the EPRI Cable Wrap Fire Barrier Tailored Collaboration assumes that the fire compartment temperature will follow the ASTM E-119 time-temperature relationship as soon as the compartment hot gas layer reaches $1000^{\circ} \mathrm{F}$.

BASIS:

- The Handbook (Ref. pg. 6-75) states that "Flashover of an enclosure is likely to occur if the temperature of the upper gas layer reaches approximately $1000^{\circ} \mathrm{F}$ $\left(600^{\circ} \mathrm{C}\right.$ ).

The EPRI assumption of $1000^{\circ} \mathrm{F}$ is therefore conservative.

- As stated near the top of page 6-67 of the Handbook, the intensity of the fire will be somewhat lower when the walls and ceiling absorb significant amounts of energy rather than when act as insulation or radiation barriers.

Since power plant compartments are typically concrete barriers which absorb significant amounts of heat, the intensity of fire will tend to be less severe than fires in compartments with insulated barriers.

- ASTM E-119 has been the accepted standard for testing fire barrier systems since the early $1900^{\prime}$ s. The standard, which was developed from actual fire tests, has been used to evaluate the fire endurance capabilities of fire barriers since it was developed. The time-temperature relationship represented by the curve represents a fully-involved fire compartment.
- According to Fred Mowrer of the University of Maryland, plume and ceiling effects disappear in a fully involved fire compartment as the plume/ceiling jet gas are no longer more buoyant than surrounding gases.
- Use of the ASTM E-119 time-temperature relationship to represent a fully involved power plant compartment fire is therefore consistent with industry practice. Assuming that that occurs when the hot gas layer reaches $1000^{\circ} \mathrm{F}$ is more conservative than the $1100^{\circ} \mathrm{F}$ noted in the Handbook.

Reference: NFPA Fire Protection Handbook, Seventeenth Edition, Section 6/Chapter 6.

## Appendix C

## Total Heat Load Results

Figure $\mathrm{C}-1$

Total Heat Load


Total Heat Lood

| Scenario | 2 | Subconfig. |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | Target | Hot Gas Layer | ASTM E-119 |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btu/s-sf) | (1000Btu/sf) | (10008tu/st) | (10008tu/st) |
| 0 | 100 | 0.4 | 0.0 | 0.00 | 0.00 |
| 5 | 111 | 0.2 | 0.09 | 0.09 | 0.34 |
| 10 | 123 | 0.3 | 0.17 | 0.17 | 1.37 |
| 15 | 136 | 0.3 | 0.25 | 0.25 | 2.94 |
| 16 | 138 | 0.3 | 0.27 | 0.27 | 3.30 |
| 25 | 167 | 0.4 | 0.44 | 0.44 | 6.88 |
| 30 | 184 | 0.4 | 0.55 | 0.55 | 9.17 |
| 35 | 203 | 0.3 | 0.66 | 0.66 | 11.62 |
| 40 | 224 | 0.3 | 0.76 | 0.76 | 14.24 |
| 50 | 270 | 0.5 | 1.00 | 1.00 | 19.87 |
| 60 | 322 | 0.5 | 1.27 | 1.27 | 25.97 |


| Scenario 3 Subconfig. 1 |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | Target | Hot Gas Layer | ASTM E-119 |
| Time | Temperature | Inc Heat Fiux | Total Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btu/s-sf) | (10008tu/sf) | (10008tu/sf) | (10008tu/st) |
| 0 | 613 | 1.0 | 0.0 | 0.00 | 0.00 |
| 5 | 690 | 1.3 | 0.35 | 0.11 | 0.34 |
| 10 | 778 | 1.6 | 0.79 | 0.23 | 1.37 |
| 15 | 879 | 2.1 | 1.34 | 0.38 | 2.94 |
| 20 | 993 | 2.7 | 2.06 | 0.58 | 4.81 |
| 22 | 1044 | 3.0 | 2.41 | 0.68 | 5.60 |
| 30 | 1278 | 5.1 | 4.34 | 1.25 | 9.10 |
| 31 | 1278 | 5.1 | 4.64 | 1.34 | 9.57 |
| 40 | 1278 | 5.1 | 7.38 | 2.19 | 14.13 |
| 50 | 1278 | 5.1 | 10.41 | 3.15 | 19.76 |
| 60 | 1278 | 5.1 | 13.44 | 4.14 | 25.87 |

Total Heat Load
Scenario 1
Subconfig. 2

|  |  |  | Target | Hot Gas Layer | ASTM E-119 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Time | Temperature | inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| $(\mathrm{min})$ | $(F$ | (Btu/s-sf) | $(1000 \mathrm{Btu} / \mathrm{sf})$ | $(1000 \mathrm{Btu} / \mathrm{sf})$ | $(1000 \mathrm{Btu} / \mathrm{st})$ |
| 0 | 1600 | 9.5 | 0.0 | 0.00 | 0.00 |
| 6 | 1600 | 9.5 | 3.41 | 0.11 | 0.48 |
| 12 | 1600 | 9.5 | 6.82 | 0.22 | 1.87 |
| 18 | 1600 | 9.5 | 10.23 | 0.35 | 3.92 |
| 24 | 1600 | 9.5 | 13.64 | 0.48 | 634 |
| 30 | 1600 | 9.5 | 17.05 | 0.63 | 9.05 |
| 36 | 1600 | 9.5 | 20.46 | 0.80 | 12.02 |
| 42 | 1600 | 9.5 | 23.86 | 0.99 | 15.21 |
| 48 | 1600 | 9.5 | 27.27 | 1.23 | 18.58 |
| 54 | 1600 | 9.5 | 30.68 | 1.55 | 22.14 |
| 60 | 1600 | 9.5 | 34.09 | 1.96 | 25.86 |

Total Heat Load
Scenario 2


Total Heat Load

| enario 3 Subconfig. 2 |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | Target | Hot Gas Layer | ASTM E-119 |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btu/s-sf) | (1000Btw/sf) | (10008tu/st) | (1000Btu/st) |
| 0 | 1600 | 9.5 | 0.0 | 0.00 | 0.00 |
| 5 | 1600 | 9.5 | 2.84 | 0.09 | 0.34 |
| 10 | 1600 | 9.5 | 5.68 | 0.16 | 1.37 |
| 15 | 1600 | 9.5 | 8.52 | 0.24 | 2.94 |
| 16 | 1600 | 9.5 | 9.09 | 0.25 | 3.30 |
| 25 | 1600 | 9.5 | 14.20 | 0.40 | 6.88 |
| 30 | 1600 | 9.5 | 17.05 | 0.49 | 9.17 |
| 35 | 1600 | 9.5 | 19.89 | 0.59 | 11.62 |
| 40 | 1600 | 9.5 | 22.73 | 0.70 | 14.24 |
| 50 | 1600 | 9.5 | 28.41 | 0.91 | 19.87 |
| 60 | 1600 | 9.5 | 34.09 | 1.14 | 25.97 |

Total Heat Load

| Scenario | 4 | Subconfig. |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | Target | Hot Gas Layer | ASTM E-119 |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btu/s-sf) | (1000Btw/sf) | (1000Btu/5t) | (10008tu/st) |
| 0 | 964 | 2.5 | 0.0 | 0.00 | 0.00 |
| 1 | 965 | 2.5 | 0.15 | 0.02 | 0.00 |
| 2 | 102 | 0.2 | 0.23 | 0.03 | 0.02 |
| 10 | 102 | 0.2 | 0.34 | 0.14 | 1.23 |
| 15 | 102 | 0.2 | 0.41 | 0.21 | 2.80 |
| 25 | 102 | 0.2 | 0.55 | 0.35 | 6.73 |
| 30 | 102 | 0.2 | 0.62 | 0.42 | 9.02 |
| 35 | 102 | 0.2 | 0.69 | 0.48 | 11.47 |
| 40 | 102 | 0.2 | 0.76 | 0.55 | 14.09 |
| 50 | 102 | 0.2 | 0.89 | 0.69 | 19.72 |
| 60 | 102 | 0.2 | 1.03 | 0.83 | 25.83 |

## TR 102 <br> APPENDIX E

## Using FIVE Fire Modeling to Approximate Exposure TimeHistory for Thermo-Lag Configurations in OC Fire Zone TB-FZ-11C

Letter Report
September 26, 1995

Prepared by:
GPUN Risk Analysis Group
Using FIVE Fire Modeling to Approximate Exposure Time-History for Thermo-Lag Configurations

## 1. PURPOSE

The purpose of this report is to document the detailed fire modeling performed for fire scenarios involving Thermo-Lag wrapped condult and armored cable wrapped in Thermo-Lag, as identified during plant walkdowns of OC fire zone TB-FZ-11C. Thermo-Lag exposure timehistory results were obtained for each T-L configuration determined to be impacted by a fire source/combustible.

## 2. APPROACH

The approach followed for modeling of fire scenarios is based on methodology detailed in the EPRI "Methods for Evaluation of Cable Wrap Fire Barrier Performance" document (Ref. 1) which was based on fire modeling techniques developed for FIVE (Ref. 2). Fire modeling was accomplished in two steps:

1. Screening walkdowns to eliminate from further consideration those fire ignition sources and potential fire scenarios that cannoi develop into damaging fires.
2. Fire modeling of spectic scenarios that were not screened out in the first step.

For each Thermo-Lag configuration analyzed, one or more fire scenarios fincluding bounding scenarios) were postulated based on the following:
a. information gathered during plant walkdowns (l.e., fixed or transient ignition source and Thermo-Lag target location, type and quantity of combustibles present in zone);
b. information obtained from reviewing applicable sections in Appendix R docurients for OC (Ref. 3);
c. information obtained from reviewing OC procedure 120.5, Rev. 5, "Control of Combustibles." (Ref. 4)
d. informati in provided by GPUN engineers (specializing in fire protection, risk assessment, mechanical).

Thermo-Lag FIVE analyses were performed using computerized Excel spreadsheets which duplicate FIVE Worksheets 1,2 , or 3 and are linked to tables containing the same information as in FIVE Reference Tables. For each Thermo-Lag fire modeling scenario, results are presented as exposure time-temperature profiles. In all scenarios evaluated, time-dependenit tempeiatures were derived by incrementally adding the total heat content of the fuel into the fire compartment hot gas layer.

## 3. FIRE MODELING ASSUMPTIONS

The following Thermo-Lag fire modeling assumptions were made:

1. Worst-case fixed fire source scenarios were modeled for each Thermo-Lag subconfiguration. The determination of which fixed fire source might cause the worst-
case Thermo-Lag fire scenario was made during the plant fire area walkdown and was based on minimum damage threshold heights and distances calculated for typical fixed fire sources (e.g., electrical cabinets, electrical motors, lube oil in pumps) found at Oyster Creek.
2. Due to the location of the T-L wrap in this fire zone (on top of switchgears 1C \& 1D), no transient combustible scenarios were noted during walkdowns or postulated as plausible.
3. To ensure conservative results, worst-case fire plume (or ceiling jet) scenarios and/or radiant flux scenario were first modeied. For T-L sub-configurations not in the plume or ceiling jet, hot gas layer fire scenarios were evaluated.
4. Electrical cable inside cabinets, motor control centers, switchgear and cable trays was assumed to be non-qualified (i.e., non IEEE-383 type). This assumption is conservative since electrical cable purchased and installed at Oyster Creek in the past decade was IEEE-383 type.
5. The Fire Hazard Tool developed for the EPRI Cable Wrap Fire Barrier Tailored Collaboration assumes that the fire compartment temperature will follow the ASTM E-119 time-temperature curve as soon as the compartment hot gas layer (HGL) reaches $1000^{\circ} \mathrm{F}$. When the HGL temperature reaches $1000^{\circ} \mathrm{F}$, the plume (or ceiling jet) temperatures are assumed to follow that of the E-119 (see discussion in Appendix B).
6. The Thermo-Lag wrap is assumed to burn by itself (i.e., even after the initial fire source is out of fuel) unless otherwise stated. Per, the EPRI Method (Ref. 1, Appendix I-D), credit may be taken for a time lag before the Thermo-Lag ignites based on exposure temperature.
7. The maximum plume and ceiling jet temperature is assumed to be $1600^{\circ} \mathrm{F}$, which corresponds to flame temperature. This assumption is valid until hot gas layer temperature exceeds $1600^{\circ} \mathrm{F}$, at which point the plume temperature is assumed to be equal to hot gas layer temperature.

## 4. FIRE MODELING RESULTS

### 4.1 Fire Zone TB-FZ-11C

Fire zone TB-FZ-11C is the Turbine Building Switchgear Room, West End of the Mezzanine Level. There is only one area in this fire zone that contains the 3-hr rated fire barrier T-L wrap condult. This T-L wrap conduit sits on top of swithgears 1C \& 1D and it is designated as location No. 13 on the general arrangement drawing shown in Appendix D.

This fire zone has unprotected openings through the floor to fire zone TB-FZ-11D and through the walls except the north wall. The north wall adjoins the fire areas TB-FA-3A and TB-FA-3B which contain safety related 4160 V switchgears 1C \& 1D and are separated from this zone by 3-hour fire rated roll-up doors. Also, Fire area TB-FA-26, Battery Room, is enveloped with three hour fire resistive rated barriers within this zone.

Due to the location of the T-L wrap in this fire zone (on top of switchgears 1C \& 1D), no transient combustible scenarios were noted during walkdowns or postulated as plausible. Based on fire
modeling experience for other fire zones, even a quart of oil spill later ignited lasts less than one minute (see Fire Zone RB-FZ-1E, Location No. 10, Scenario \# 1). Ref. 5, Attachment 3, Table 13 indicates that even at maximum plume temperature of $1600^{\circ} \mathrm{F}$, it takes about one minute for the Thermo-lag to reach the ignition temperature. Therefore, this transient combustible scenario will be insignificant. However, two fixed ignition sources were modeled for this zone. It should alse be noted that even though, there is hydrogen seal oil in this fire zone, it is only in piping that passes through the zone.

## Scenario 1

This fire scenario models non-safety related switchgear 1B self-igniting. The T-L wrap is not in the plume of such fire and would only see the hot gas layer region. However, there are three 24 inch wide cable trays stacked on top of each other 1 ft . apart in the plume of such fire (directly over the back vents of the switchgear 1 B , about 4 ft . above, and run paraliel to the west wall along the back of the switchgear). It is assumed that 20 ft of the first cable tray immediately above the switchgear 1B ignites (being in the plume of the switchgear fire). This subsequently leads to ignition of 22 ft . and 24 ft . of the other cable trays above. The time required for these sequential fires to occur is conservatively ignored here. In addition to the vertical propagation, the fire is also assumed to propagate horizontally along the exposed cable tray with a flame speed of 10 feet per hour (per Ref. 2, Appendix I-C). The total amount of cable tray assumed to be available for this fire scenario is 50 feet per tray (for 3 hour fire duration).

The heat release rates considered for this scenario are as follows (by fire source):

1. electrical panel (switchgear 1B) with nonqualified cable - $400 \mathrm{BTU} / \mathrm{sec}$ (Ref.1); this HRR is applicable to scenario time $\mathrm{t}=0-15$ minutes;
2. first cable tray - 23.4 BTU/sec- ft2 (Ref. 2, FIVE Table 1E, cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); since 40 ft are initially burning, the HRR for this source alone is $936 \mathrm{BTU} / \mathrm{sec}$; this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be 983 BTU/s, which includes the HRR from 2 ft 2 of additional cable tray);
3. second cable tray - 23.4 BTU/sec- ft2 (Ref. 2, FIVE Table 1E, cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); just as with the first cable tray, the HRR for this source alone is 1030 BTU/sec (since 44 ft 2 are initially burning); this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be $1076 \mathrm{BTU} / \mathrm{s}$, which includes the HRR from 2 ft 2 of additional cable tray);
4. third cable tray - $23.4 \mathrm{BTU} / \mathrm{sec}$ - ft 2 (Ref. 2, FIVE Table 1 E , cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); just as with the second cable tray, the HRR for this source alone is $1123 \mathrm{BTU} / \mathrm{sec}$ (since 48 ft 2 are initially burning); this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be $1170 \mathrm{BTU} / \mathrm{s}$, which includes the HRR from 2 ft 2 of additional cable tray);

The total available heat (Qtot) from each fire source is:

1. electrical panel $-360,000$ BTUs (it assumes that the panel contains enough fuel to sustain a 15 minute fire);
2. first cable tray $-20,000,000$ BTUs it assumes $10,000 \mathrm{BTU} / \mathrm{bm}$ of cable insulation (per Ref. 6, page E2-1) $\times 50$ linear feet $\times 40 \mathrm{lbm} / \mathrm{linear}$ foot (engineering judgment for $100 \%$ filled, 24 inch wide cable tray)];
3. second cable tray $-20,000,000$ BTUs [it assumes $10,000 \mathrm{BTU} / \mathrm{lbm}$ of cable insulation (per Ref. 6, page E2-1) $\times 50$ linear feet $\times 40 \mathrm{lbm} /$ linear foot (engineering judgment for $100 \%$ filled, 24 inch wide cable tray)];
4. third cable tray $-20,000,000$ BTUs [it assumes 10,000 BTU/hbm of cable insulation (per Ref. 6, page E2-1) $\times 50$ linear feet $\times 40 \mathrm{lbm} / \mathrm{linear}$ foot (engineering judgment for $100 \%$ filled, 24 inch wide cable tray)];

The floor area for this fire zone is 2,666 square feet (Ref. 3) with a height of about 20 feet (General Arrangement Drawings). It should be noted that this zone contains fire areas TB-FA-3A, TB-FA-3B, and TB-FA-26. The floor areas for TB-FA-3A and TB-FA-3B combined is estimated at about 1000 square feet and the floor area for TB-FA-26 is about 250 square feet. The height for these fire areas were estimated at about 10 feet.

## Results

The time temperature profile scenario calculated by the model are presented in Table 1. The model results indicate that hot gas layer effects in the zone due to this postulated fire cause the temperature to exceed $1000^{\circ} \mathrm{F}$ at time $=19$ minutes. It should be noted that these HGL. temperatures in this compartment are expected to be very conservative due to the number and size of the openings that exist in this fire zone and the tortuous path that the hot gas needs to take to reach the T-L wrap conduit area vs. other direct openings out of this fire zone.

Past the $1000^{\circ} \mathrm{F}$ HGL temperature, it is assumed that a fire compartment temperature wil follow the ASTM E-119 time-temperature relationship (Ref. 1). Appendix B contains the basis for this assumption. A sample model worksheet is provided in Appendix A, Figure A-1.

Table 1. Temperature-Time History for Fire Scenario 1.

| Time | HRTR to | HRR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL |  |  |  |
| (min.) | (Btu/s) | (Btu/s) | (BTU) | Target - (F) | HGL - (F) |
| 0 | 3489 | 3489 | 0 | 100 | 100 |
| 6 | 3629 | 3629 | 1305440 | 302 | 302 |
| 12 | 3769 | 3769 | 2663280 | 589 | 589 |
| 18 | 3509 | 3509 | 3926520 | 953 | 953 |
| 19 | 3532 | 3532 | 4138440 | 1025 | 1025 |
| 25 |  |  |  | $1300^{*}$ | $1300^{*}$ |
| 60 |  |  |  | $1638{ }^{*}$ | $1638^{\circ}$ |
| 90 |  |  |  | 1750* | 1750** |
| 120 |  |  |  | $1835^{*}$ | $1835{ }^{*}$ |
| 150 |  |  |  | $1862^{\text {\% }}$ | $1862^{*}$ |
| 180 |  |  |  | $190{ }^{*}$ | $1900^{*}$ |

* Temperatures follow those of ASTM E-119.


## Scenario 2

All the cable trays in this fire zone are at a higher elevation than the T-L wrapped conduit. However, this scenario models self-ignition of one of these cable trays and evaluates the timetemperature profile in the room for the case when/ff the HGL temperature reaches $1000^{\circ} \mathrm{F}$ and it becomes a fully involved fire.

The most bounding configuration for this scenario was found to be three cable trays stacked on top of each other, 1 ft . apart, and about 4 ft behind and 2 ft . above the T-L wrap conduit. The cable trays are 24 inches wide. It is assumed that 2 ft . of the bottom cable tray self-ignites and this ignites 4 ft . of cable tray in the middle, and consequently, 6 ft . of cable tray on top. It is conservatively assumed that all the cable trays ignition occur instantaneously for calculation simplification. In addition to the vertical propagation, the fire is also assumed to propagate horizontally along the exposed cable tray with a flame speed of 10 feet per hour (per Ref. 2, Appendix (-C). The total amount of cable tray assumed to be avallable for this fire scenario is 30 feet per tray (for 3 hour fire duration).

The heat release rates considered for this scenario are as follows (by fire source):

1. first cable tray - 23.4 BTU/sec- ft2 (Ret. 2, FIVE Table 1E, cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); since 4 tt2 are initially burning, the HRR for this source alone is $93.6 \mathrm{BTU} / \mathrm{sec}$; this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be 140.4 BTU/s, which
includes the HRR from 2 ft 2 of additional cable tray);
2. second cable tray - 23.4 BTU/sec- ft2 (Ref. 2, FIVE Table 1E, cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); just as with the first cable tray, the HRR for this source alone is $187 \mathrm{BTU} / \mathrm{sec}$ (since $8 \mathrm{ft2}$ are initially burning); this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be 234 BTU/s, which includes the HRR from 2 ft 2 of additional cable tray);
3. third cable tray - 23.4 BTU/sec- ft2 (Ref. 2, FNE Table 1E, cable \#5 adjusted to consider Equation 2 on FIVE manual, page 10.4-10); just as with the second cable tray, the HRR for this source alone is 281 BTU/sec (since $12 \mathrm{ft2}$ are initially burning); this HRR will increase with time, as fire propagates horizontally along the tray (i.e., 6 minutes into the fire, the HRR for this source will be 328 BTU/s, which inciudes the HRR from 2 ft 2 of additional cable tray);

The total available heat (Qtot) from each fire source is:

1. first cable tray $-12,000,000$ BTUs [t assumes $10,000 \mathrm{BTU}$ /lom of cable insulation (per Ref. 6. page E2-1) $\times 30$ linear feet $\times 40 \mathrm{lbm} /$ linear foot (engineering judgment for $100 \%$ filled, 24 inch wide cable tray)];
2. second cable tray $-12,000,000$ BTUs [it assumes $10,000 \mathrm{BTU} / \mathrm{bm}$ of cable insulation (per Ref. 6. page E2-1) $\times 30$ linear feet $\times 40 \mathrm{lbm} /$ linear foot (engineering judgment for $100 \%$ filled, 24 inch wide cable tray)];
3. third cable tray $-12,000,000$ BTUs [it assumes $10,000 \mathrm{BTU} / \mathrm{lbm}$ of cable insulation (per Ref. 6, page E2-1) $\times 30$ linear feet $\times 40 \mathrm{lbm} /$ linear foot (engineering judgment for $100 \%$ filled, 24 inch wide cable tray)];

## Results

The time temperature profile scenario calculated by the model are presented in Table 2. The model results indicate that hot gas layer effects in the zone due to this postulated fire cause the temperature to exceed $1000^{\circ} \mathrm{F}$ at time $=34$ minutes. It should be noted that these HGL temperatures in this compartment are expected to be very conservative due to the number and size of the openings that exist in this fire zone.

Past the $1000^{\circ}$ F HGL temperature, it is assumed that a fire compartment temperature will follow the ASTM E-119 time-temperature relationship (Ref. 1). Appendix B contains the basis for this assumption. A sample model worksheet is provided in Appendix A, Figure A-2.

Table 2. Temperature-Time History for Fire Scenario 2.

| Time | HRRT to | HRR to | QTOT | Temp. | Temp. |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Target | HGL |  |  |  |
| (min.) | (Bfu/s) | (Btu/s) | (BiU) | Target-(F) | HGL - (F) |
| 0 | 562 | 562 | 0 | 100 | 100 |
| 6 | 702 | 702 | 252720 | 171 | 171 |
| 12 | 842 | 842 | 555840 | 268 | 268 |
| 18 | 982 | 982 | 909360 | 400 | 400 |
| 24 | 1122 | 1122 | 1313280 | 580 | 580 |
| 30 | 1262 | 1262 | 1767600 | 828 | 828 |
| 34 | 1356 | 1356 | 2093040 | 1042 | 1042 |
| 60 |  |  |  | 1550* | $1550{ }^{\text {a }}$ |
| 90 |  |  |  | 1704* | 1704* |
| 150 |  |  |  | $1850^{*}$ | $1850^{\circ}$ |
| 180 |  |  |  | 1888* | $1888^{\circ}$ |

*Temperatures follow those of ASTM E-19.

## 5. FIRE MODELING SUMMARY

Fire modeling was performed for two fixed fire Source-Thermo-lag wrap configurations in Oyster Creek Fire Zone TB-FZ-11C. The analysis results are presented as time-temperature profiles. These results are used in the following section to develop total heat load and then convert to equivalent ASTM E-119 exposures.

## 6. APPROXIMATING EXPOSURE TIME-HISTORY OF THERMO-LAG SUBCONFIGURATIONS - THE TOTAL HEAT LOAD CONCEPT

Calculation of an exposure time-temperature profile for a specific T-L Sub-configuration is based on the methodology presented in Reference 1. The time -temperature profiles are used to deveiop the total heat load (Reference 1, Appendix III-A) for each fire exposure. The total heat load concept holds that the area under the incident heat flux vs. time curve, or total heat lead, can be used as a measure of fire severity. Comparing the total heat load of a fire scenario exposure to the total heat load of a test exposure (ASTM E-119) allows one to predict the response of a Thermo-Lag barrier to the scenario exposure. The total heat load per unit area of fire barrier surface is equal to the area under the incident heat flux-time curve, corrected for convection effects as described in Reference 1, Appendix III-A.

| Location <br> No.(GA <br> DWG, <br> App. D) | Scenaio | Description | Peak <br> Temperature at <br> Target ( | Total Heat Load <br> Equivalent E-119 <br> Exposure Duration <br> (min.) |
| :--- | :--- | :--- | :--- | :--- |
| 13 | 1 | Switchgear 1B and <br> Cable Tray | 1900 | $>150,<180$ |
| 13 | 2 | Cable Tray | 1888 | $>150,<180$ |

Appendix C contains tables with Total Heat Load values calculaied for Thermo-Lag scenarios 1 , and 2.
7. REFERENCES

1. EPRI Report "Methods for Evaluation of Cable Wrap Fire Barrier Performance", July, 1995
2. EPRI TR-100370, "Fire-Induced Vuinerability Evaluation," April 1992.
3. GPUN, "Fire Hazards Analysis Report - OC," Doc. No. 990-1746, Rev. 8, May 24, 1995.
4. GPUN - OC Procedure No. 120.5 (Revision 5), May 20, 1993.
5. Thermo-Lag 330-1 Combustibility Study, NEI, 1994.
6. GPUN - TMI-1 Procedure No. 1035 (Revision 24), August 23, 1994.

## Appendix A

## Sample Scenario Worksheets

Figure A-1

| Temperature calculation at time |  | min | 6 |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | $f 12$ | 2666 |
| 1 | Location of Target |  | Hot Gas Layer |
| 2 | Height of Target above Fire Source | $f$ | 2.00 |
| 3 | Height from Fire Source to Ceiling, H | f | 10.00 |
| 4 | Ratio of Target HeighVCeiling Height |  | 0.200 |
| 5 | Longitudinal Distance from Fire Source to Target, L | f | NA |
| 6 | Longitudinal Distance to Height Ratio, LHH |  | NA |
| 7 | Enclosure Width, W | ft | NA |
| 8 | Height to Width Ratio, H/W |  | NA |
| 9 | Peak Fire Intensity | Btu/s | 3629 |
| 10 | Fire Location Factor |  | 1 |
| 11 | Effective Fire Intensity | Biu/s | 3629 |
| 12 | Plume Temperature Rise at Target | F | 1600 |
| 13 | Temp Rise Factor at Target |  | 0.00 |
| 14 | Temperature Rise at Target | F | 0 |
| 15 | Temperature at Target | F | 302 |
| 16 | Temperature in HGL | $F$ | 302 |
| 17 | Qtot to HGL | Btu | 1306440 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated Qnet | Btu | 78386 |
| 20 | Calculated Enclosure Volume, V | $f 13$ | 26660 |
| 21 | Calculated Qnet/V | Qnevfl 3 | 2.94 |
|  | HGL Temperature Increase | $F$ | 202 |
|  |  |  |  |

Figure A-2

| Temperature calculation at time |  | min | $\bigcirc$ |
| :---: | :---: | :---: | :---: |
| Maximum ambient temperature |  | F | 100 |
| Floor Area |  | $f 2$ | 2686 |
| 1 | Location of Target |  | Hot Gas Layer |
| 2 | Height of Target above Fire Source | fi | 0.10 |
| 3 | Height from Fire Source to Celling, H | fif | 5.00 |
| 4 | Ratio of Target Height/Ceiling Height |  | 0.020 |
| 5 | Longitudinal Distance from Fire Source to Target, L | fi | NA |
| 6 | Longitudinal Distance to Height Ratio, L/H |  | NA |
| 7 | Enclosure Width, W | $f$ | NA |
| 8 | Height to Width Ratio, HWW |  | NA |
| 9 | Peak Fire Intensity | Btu/s | 702 |
| 10 | Fire Location Factor |  | 1 |
| 11 | Effective Fire Intensity | Btu/s | 702 |
| 12 | Plume Temperature Rise at Target | F | 1600 |
| 13 | Temp Rise Factor at Target |  | 0.00 |
| 14 | Temperature Rise at Target | F | 0 |
| 15 | Temperature at Target | F | 171 |
| 16 | Temperature in HGL | $F$ | 171 |
| 17 | Qtot to HGL | Btu | 252720 |
| 18 | Estimated Heat Loss Fraction |  | 0.94 |
| 19 | Calculated Qnet | Btu | 15163 |
| 20 | Calculated Enclosure Volume, V | $f 3$ | 13330 |
| 21 | Calculated QnetV | QneU/t 3 | 1.14 |
|  | HGL Temperature Increase | F | 71 |
|  |  |  |  |

## Appendlx B

Basis for Following ASTM E-119 Time-Temperature Relationshipo When T(HGL) is $1000{ }^{\circ} \mathrm{F}$

## TRANSITION TO

## ASTM E-119 TIME-HISTORY

IN THE HOT GAS LAYER

ISSUE:

The Fire Hazard Tool currently under development in the EPRI Cable Wrap Fire Barrier Tailored Collaboration assumes that the fire compartment temperature will follow the ASTM E119 time-temperature relationship as soon as the compartment hot gas layer reaches $1000^{\circ} \mathrm{F}$.

## BASIS:

The Handbook (Ref., pg. 6-75) states that "Flashover of an enclosure is likely to occur if the temperature of the upper gas layer reaches approximately $1000^{\circ} \mathrm{F}\left(600^{\circ} \mathrm{C}\right)$.

The EPRI assumption of $1000^{\circ} \mathrm{F}$ is therefore conservative.

As stated near the top of page 6-67 of the Handbook, the intensity of the fire will be somewhat lower when the walls and ceiling absorb significant amounts of energy rather than when act as insulation or radiation barriers.

Since power plant compartments are typically concrete barriers which absorb significant amounts of heat, the intensity of fire will tend to be less severe than fires in compartments with insulated barriers.

ASTM E-119 has been the accepted standard for testing fire barrier systems since the early 1900 's. The standard, which was developed from actual fire tests, has been used to evaluate the fire endurance capabilities of fire barriers since it was developed. The timetemperature relationship represented by the curve represents a fully-involved fire compartment.

According to Fred Mowrer of the University of Maryland, plume and ceiling effects disappear in a fully involved fire compartment as the plume/ceiling jet gas are no longer more buoyant than surrounding gases.

Use of the ASTM E-119 time-temperature relationship to represent a fully involved power plant compartment fire is therefore consistent with industry practice. Assuming that that occurs when the hot gas layer reaches $1000^{\circ} \mathrm{F}$ is more conservative than the $1100^{\circ} \mathrm{F}$ noted in the Handbook.

Reference: NFPA Fire Protection Handbook, Seventeenth Edition, Section 6/Chapter 6.

## Appendix C

## Total Heat Load Results

Figure C-1

Total Heat Load

|  |  |  | Target | Hot Gas Layer | ASTM E-118 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Time | Temperature | Inc Heat Flux | Totai Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btws-sf) | (1000Btu/s) | (1000Btu/s) | (1000Btu/sf) |
| 0 | 100 | 0.4 | 0.0 | 0.00 | 0.00 |
| 6 | 302 | 0.4 | 0.14 | 0.14 | 0.41 |
| 12 | 589 | 1.0 | 0.40 | 0.40 | 1.65 |
| 18 | 953 | 2.4 | 1.02 | 1.02 | 3.56 |
| 18 | 1025 | 2.9 | 1.18 | 1.18 | 3.82 |
| 25 | 1300 | 5.3 | 2.65 | 2.65 | 6.31 |
| 60 | 1638 | 10.1 | 18.81 | 18.81 | 25.06 |
| 90 | 1750 | 12.3 | 38.83 | 38.83 | 45.76 |
| 120 | 1835 | 14.2 | 62.74 | 62.74 | 69.38 |
| 150 | 1862 | 14.8 | 88.85 | 88.85 | 85.02 |
| 180 | 1800 | 15.8 | 116.46 | 116.46 | 122.34 |

Figure C-2

Total Heat Load

| Scenario 2 |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | Target | Hot Gas Layer | ASTM E-119 |
| Time | Temperature | Inc Heat Flux | Total Heat Load | Total Heat Load | Total Heat Load |
| (min) | (F) | (Btw/s-s? | (1000Btw/s) | (1000Btu/sf) | (1000Btu/s) |
| 0 | 100 | 0.4 | 0.0 | 0.00 | 0.00 |
| 6 | 171 | 0.4 | 0.13 | 0.13 | 0.41 |
| 12 | 268 | 0.4 | 6.28 | 0.28 | 1.65 |
| 18 | 400 | 0.7 | 0.47 | 0.47 | 3.56 |
| 24 | 580 | 1.0 | 0.77 | 0.77 | 5.93 |
| 30 | 828 | 1.8 | 1.27 | 1.27 | 8.64 |
| 34 | 1042 | 3.0 | 1.84 | 1.84 | 10.54 |
| 60 | 1550 | 8.6 | 10.87 | 10.87 | 24.95 |
| 90 | 1704 | 11.3 | 28.78 | 28.76 | 45.65 |
| 150 | 1850 | 14.5 | 75.25 | 75.25 | 84.54 |
| 180 | 1888 | 15.5 | 102.30 | 102.30 | 121.87 |

Appendix D

## Ceneral Arrangement Drawings



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