

# COMBUSTION ENGINEERING

NUCLEAR POWER SYSTEMS  
COMBUSTION ENGINEERING, INC.  
COMPONENT SERVICES  
Chattanooga, Tennessee

0-0000

WATERFORD UNIT NO: 3  
STEAM GENERATOR MANWAYS

CENC-1805

\_\_\_\_\_  
CALCULATION NO.

74270

\_\_\_\_\_  
CONTRACT NO.

LOUISIANA POWER AND LIGHT  
\_\_\_\_\_  
CUSTOMER

STEAM GENERATOR  
\_\_\_\_\_  
COMPONENT

J. E. Roberts Structural Engineer 3-14-88  
PREPARED TITLE DATE

R. B. Hale Structural Engineer 3/15/88  
VERIFIED TITLE DATE

P. L. Anderson Supervisor 3/15/88  
REVIEWED TITLE DATE

W. J. Hale Manager 3/15/88  
APPROVED TITLE DATE









DESIGNED <i>Robert</i>	3-7-88 DATE	CONSTRUCTION ENGINEERING	SHEET 5 OF 6
CHECKED <i>Hale</i>	3-15-88 DATE		CALL NO. CENC-1805
TITLE		COMPONENT ENGINEERING	CONTRACT NO. 74670
WATERFORD UNIT No. 3 STEAM GENERATOR MANWAYS			

4. ANALYSIS

B. PRIMARY MANWAY REINFORCEMENT (REF. 1 SHEET A-4E)

$A_1 = 89.769 \text{ in}^2$   
 $A_2 = 8.327 \text{ in}^2$

$A_{\text{HOLE}} = 3.125 (1.625) = 5.078 \text{ in}^2$

$A_{\text{AVAILABLE}} = A_1 + A_2 - A_{\text{HOLE}} = 93.013 \text{ in}^2 > A_{\text{REQD}} = 31.906 \text{ in}^2$

THE REINFORCEMENT REQUIREMENT IS MET BASED ON USING A 1 1/2" HELICOIL INSERT.

C. SECONDARY MANWAY REINFORCEMENT (REF. 1 SHEET A-4B)

$A_1 = 1.596 \text{ in}^2$   
 $A_2 = 10.370 \text{ in}^2$   
 $A_3 = 4.396 \text{ in}^2$

$A_2 = (4.24)(\text{CN} - \text{TRN}) - 1.625(3) - (1.032 - \text{CRN})(.375)$   
 $A_2 = (4.24)(8.561) - 4.875 - (1.630)(.375) = 31.187 \text{ in}^2$

AVAILABLE  $A = A_1 + A_2 + A_3 + A_4 = 47.499 \text{ in}^2 > A_{\text{REQD}} = 43.452 \text{ in}^2$

THE REINFORCEMENT REQUIREMENT IS MET BASED ON USING A 1 1/2" HELICOIL INSERT.

D. SHEAR STRENGTH OF HELICOIL / STUD - PRIMARY MANWAY HELICOIL SPECIFICATIONS

- MATERIAL: SA-479 TYPE 304 REF. 4
- $S_m$ : 16.2 ksi @ 650 REF. 2
- SIZE: 1 1/2" - BN x 2 1/2" REF. 3

STRENGTH OF THE STUD

TENSILE STRESS AREA OF STUD BASED ON SHANK DIAMETER  $d$

$A = \frac{\pi d^2}{4} = \frac{\pi (1.31)^2}{4} = 1.348 \text{ in}^2$

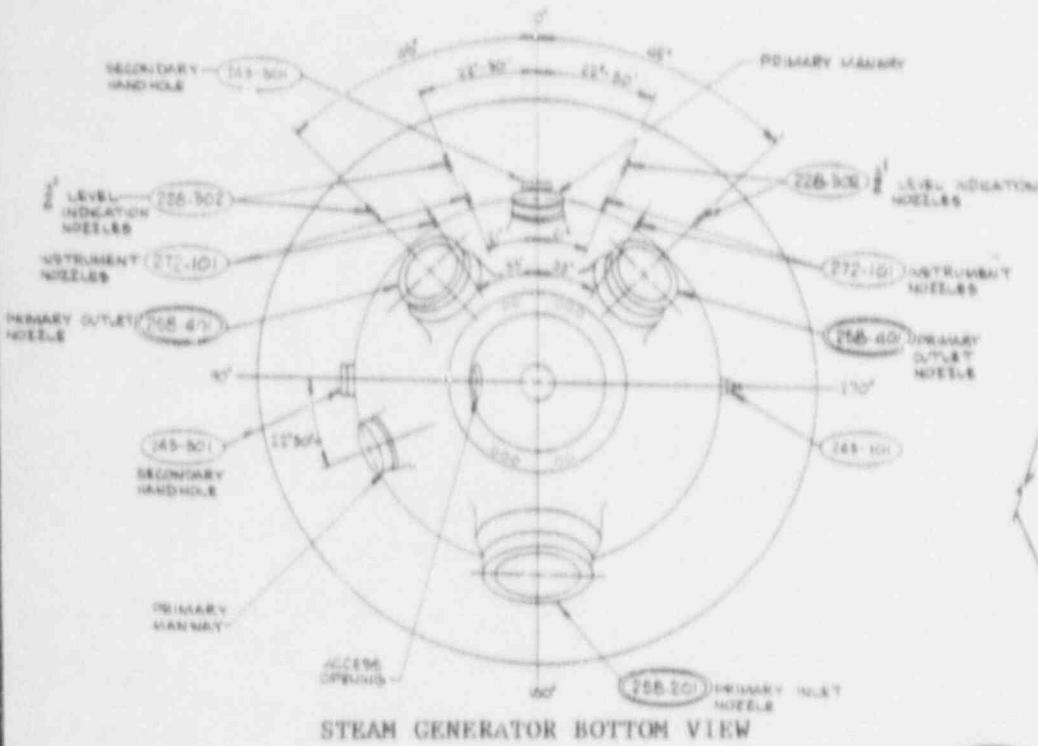
MAXIMUM DESIGN STUD LOAD AT 650°F

$F = A S_m = 1.348 (34.8) = 46.91 \text{ KIPE}$

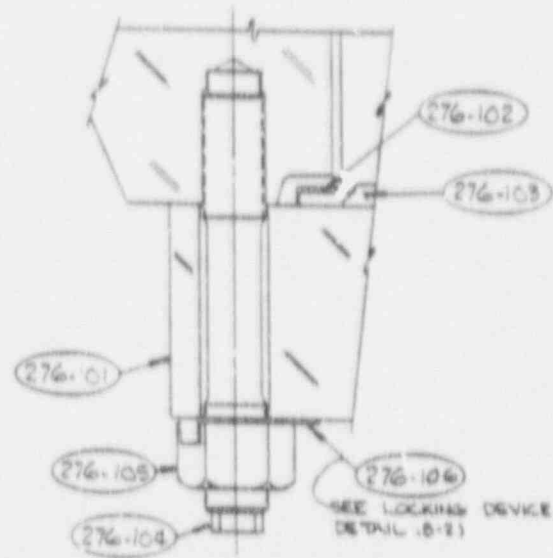
REV.	DATE	BY	CHECK	REV.	DATE	BY	CHECK	REV.	DATE	BY	CHECK



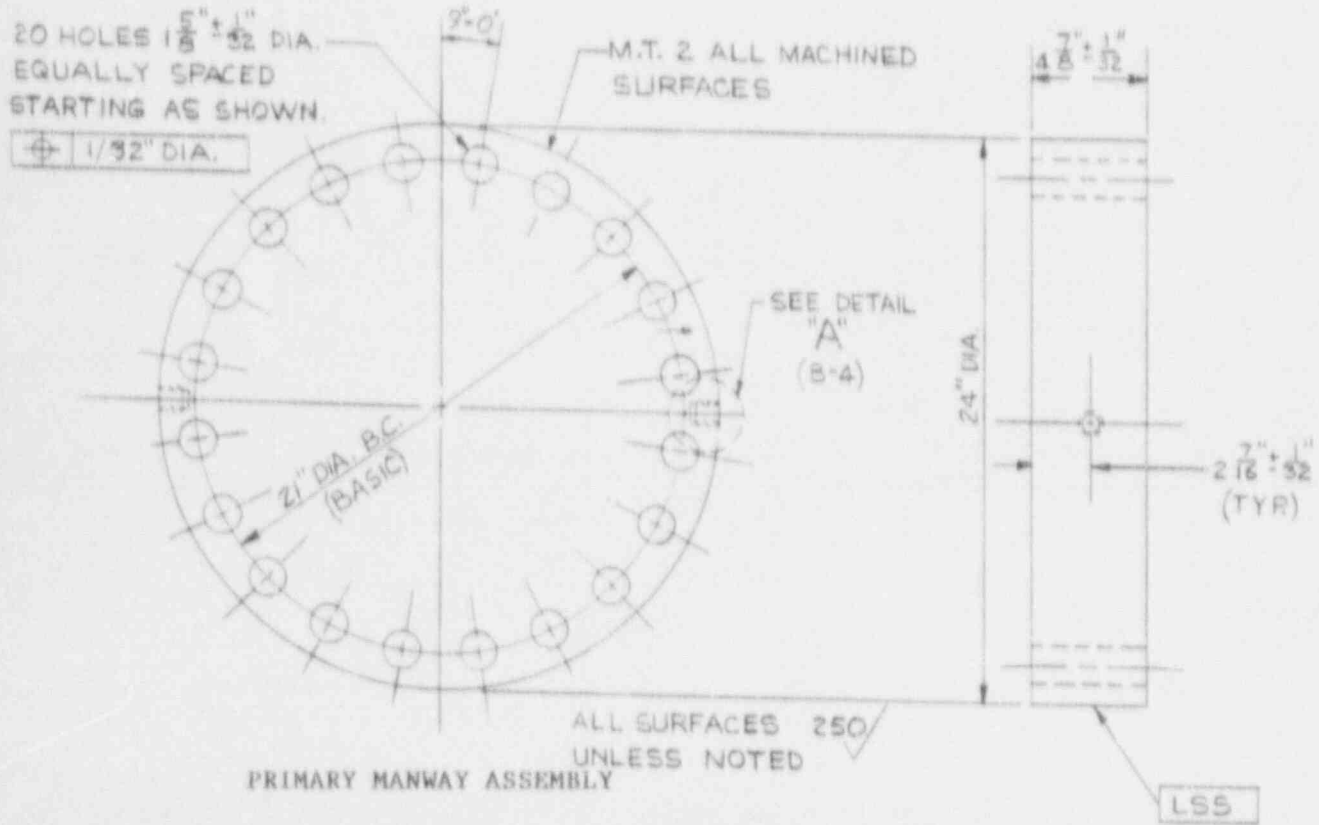
STEAM GENERATOR PRIMARY MANWAY CLOSURE DETAILS



STEAM GENERATOR BOTTOM VIEW



PRIMARY MANWAY CLOSURE



PRIMARY MANWAY ASSEMBLY



Waterford 3  
Steam Generator Design Data

Application:	Steam Generator Primary Manway Closure, 20 Studs per Closure
Oper. Pressure/Temperature:	2250 psia/611°F (hot leg)/553°F (cold leg)
Design Pressure/Temperature:	2500 psia/650°F
Stud Size:	1-1/2" - 8N - 2A
Stud Material:	SA-540, Grade B24, Class 3 (Sy) min. yield 130 ksi
Manway Material:	SA-533, Class 2 (Sy) min. yield 50 ksi
Helical Material:	SA-479, Type 304 stainless (Sy) min. yield 150 ksi (after cold working)
Max. Stud Service Stress Intensity, Sm: (from preload, pressure, and operating transients)	36.7 ksi @ 611°F operating temperature 36.7 ksi is less than 71.4 ksi  NOTE: ASME Section III allowable at 611°F is 71.4 ksi from $2 \times S_m = 71.4$ ksi ( $S_m$ Design Value = 35.7)

Sy = Yield Stress  
Sm = Stress intensity  
KSI = Kips per in<sup>2</sup>