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UNITED STATES OF AMERICA
NUCLEAR REGULATORY COMMISSION

BEFORE THE ATOMIC SAFETY AND LICENSING BOARD

In the Matter of	
) Docket Nos. 50-445 and
TEXAS UTILITIES ELECTRIC COMPANY, et al.	50-446
) (Application for
(Comanche Peak Steam Electric Station, Units 1 and 2)) Operating Licenses)

AFFIDAVIT OF JOHN C. FINNERAN, JR., ROBERT C. IOTTI AND R. PETER DEUBLER REGARDING DESIGN OF RICHMOND INSERTS AND THEIR APPLICATION TO SUPPORT DESIGN

We, John C. Finneran, Jr., Robert C. Iotti, and R. Peter Deubler, being first duly sworn hereby depose and state as follows:1

(Finneran) I am the Pipe Support Engineer for the Pipe Support Engineering Group at Comanche Peak Steam Electric Station. In this position, I oversee the design work of all pipe support design organizations for Comanche Peak. I have previously provided testimony in this proceeding. A statement of my professional and educational qualifications was received into evidence as Applicants' Exhibit 142B.

Except as otherwise indicated, each Affiant attests to all parts of this affidavit.

(Iotti) I am the Chief Engineer, Applied Physics for Ebasco Services, Inc. I have been retained by Texas Utilities Electric Company to oversee the assessment of allegations regarding the design of piping and supports at Comanche Peak Steam Electric Station ("CPSES"). A statement of my educational and professional qualifications is attached to Applicants' letter of May 16, 1984 to the Licensing Board.

(Deubler) I am the Project Manager for the Comanche Peak
Project and formerly Director of Engineering for NPS Industries,
Inc. In this position, I oversee the design work of Nuclear
Power Services on Comanche Peak including work related to the
Richmond inserts. A statement of my professional and educational
qualifications is submitted as Attachment G.

- Q. What is the purpose of this Affidavit?
- A. This Affidavit responds to six CASE allegations (see CASE's Proposed Findings at Setion VIII) and two Board concerns (see Board Memorandum and Order of December 28, 1983 at 60-66) regarding the design of Richmond inserts. In addition, this Affidavit provides information in compliance with Items 10 and 11 of Applicants' Plan to Respond to Memorandum and Order (Quality Assurance for Design) ("Applicants' Plan") (February 3, 1984). CASE's six specific allegations are related to (1) the factor of safety used for Richmond inserts, (2) testing of Richmond inserts, (3) ability to resist axial torsion, (4) methods used to analyze connections, (5) bending moments in the bolts, and (6)

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sharing of shear loads. Each item is addressed in the following sections of this Affidavit. In responding to CASE's concerns regarding items (1), (2) and (3) above, Applicants also address the two Board concerns and provide the information to comply with Applicants' Plan.

I. and II. FACTOR OF SAFETY USED FOR RICHMOND INSERTS AND TESTS

- Q. Please state the concerns raised regarding the factor of safety used for Richmond inserts and associated testing.
- A. This issue deals with a concern set forth in the Special Investigation Team's ("SIT") Report² that Applicants had employed a safety factor of 2 for Richmond inserts instead of the manufacturer's recommended value of 3.

The SIT and Board's concern is expressed in the Staff's Proposed Findings of Fact and Conclusions of Law (August 30, 1984) at 37-39. The two key issues regarding this area are (1) the appropriateness of Applicants' use of a safety factor which is lower than that recommended by the manufacturer, and (2) the lack of certain test data regarding Richmond inserts.

A. Factors of Safety

Q. Describe your evaluation of the safety factor used by Applicants as compared to that recommended by the manufacturer.

NRC Inspection Report 50-445/82-26; 50-446/82-14 dated 2/15/83 at 17-23.

A. In the manufacturer's literature regarding Richmond inserts, based on testing the manufacturer specifies the ultimate loads associated with the various sized inserts. In addition, the manufacturer selects a factor of safety and back-calculates the corresponding allowable loads, i.e., the ultimate load divided by the safety factor is equal to the allowable load. It should be noted that this factor of safety and corresponding recommended allowable loads specified by the manufacturer apply only to the Richmond insert itself and not to the threaded rod (sometimes used. interchangeably with bolt) which may be procured separately. Allowables for the threaded rod are those set forth in appropriate AISC Codes, e.g., for A-36 threaded rod the allowed load in shear is 17.7 kips.

In its design calculations, Applicants used higher allowable loads for the inserts than specified by the manufacturer. Accordingly, if the ultimate loads recommended by the manufacturer were applicable to Applicants' use of the inserts at CPSES, it could be viewed that Applicants had reduced the factor of safety recommended by the manufacturer. However, this is not the case. As set forth more fully below, taking into consideration all relevant factors (e.g., the differences between the conditions of the tests from which the Richmond insert manufacturer obtained its recommended ultimate loads and the conditions known by Applicants to exist in the actual

applications of the Richmond inserts at CPSES), the ultimate loads for the inserts are much higher than specified by the manufacturer, and the actual safety margins used by Applicants are essentially equivalent to those used by manufacturer.

The current allowable recommended loads for the inserts by the Richmond Screw Anchor Co. are based on tests conducted at the Polytechnic Institute of Brooklyn in 1957. Richmond's recommended allowable (working) loads are based on the average ultimate test loads divided by a factor of safety which has varied over the years. Tests were conducted for 3/4, 1, and 1-1/4 inch diameter inserts in shear and 1 and 1-1/2 inch diameter inserts in tension. (However, at issue at CPSES are 1 inch and 1-1/2 inch inserts.)

psi, while for the tension tests the concrete strength was 2850 psi for the 1-inch diameter insert and 2950 psi for the 1-1/2 inch diameter insert. Data from the manufacturer's tests reflect that failure in all insert shear tests and the 1-1/2 inch insert tension tests occurred due to failure of the testing anchor stud bolt. Failure in the 1 inch tension test occurred due to failure of the testing anchor stud bolt insert by concrete cone pullout. It should be noted that failure of the insert can generally be equated with failure in the concrete resulting in a cone of concrete being pulled out ("concrete cone

pullout"3.) Table A specifies the manufacturer's recommended allowable loads, and in parentheses the associated factor of safety for each relevant size insert, as they evolved over the years.

TABLE A
Recommended Allowable Loads in Kips (Factor of Safety)

Richmond Bulletin	Shear		Tension		
	1"	1-1/2"	1"	1-1/2"	
#6,1961			10.0 (2.3)	25 (2.6)	
#6,1971 #6,1975	8.0 ^{XX,+} (3.0)	18* (3.0)	10.0 (2.3) 8.27 (3.0)	25 (2,6) 21.67 (3.0)	

^{*} Estimated (apparently unsupported by manufacturer's tests)

From the foregoing, it can be seen that the failure modes of concern are either failure of the insert through concrete cone pullout or failure of the threaded rod or bolt used with the insert. As noted above, allowable loads and

⁺ Failure occurred in the testing anchor stud bolt

⁺⁺ Failure occurred due to concrete cone pull-out

^{XX} Ultimate shear load was in excess of 27,000 lbs., hence allowable could be 9.0 kips

Even if failure by internal damage of the insert occurs instead of concrete cone pullout, the load at which it occurs is essentially the same at which concrete cone pullout would occur (see the results of the March 1984 tests set forth in Attachment B).

factors of safety concerning the threaded rods used with the inserts are established by Code, adhered to by Applicants and not an issue here.

The major factor affecting cone pullout is the strength of the concrete in which the inserts are placed. Significantly, the manufacturer's tests were conducted with concrete which had a strength of between 2850 and 3220 psi (approximately 3000 psi). While the concrete at CPSES is designed for 4000 psi, it actually ranges from 4500 to above 5000 psi. We believe that the additional strength of the concrete results in a much higher ultimate failure load. Accordingly, it was Applicants' position that use of allowable loads higher than recommended by the manufacturer was justified based on the higher ultimate loads for the particular circumstances at CPSES, and the safety factor specified by the manufacturers would be essentially met.

- Q. Have there been any analyses which verify the appropriateness of Applicants' position?
- A. Yes. First we would like to discuss the safety factors in tension. The basis for Applicants' position that the ultimate load is much higher than established by the manufacturer's test has been verified by a simple comparison with the manufacturer's test results. The mechanism of tensile failure of Richmond inserts and concrete cone pullout is no doubt a complex mechanism difficult to

precisely analyze. However, the increase in the ultimate insert tensile capacity due to greater strength concrete can be conservatively calculated using the following equation:4

 $T = 4 \emptyset (f_c)^{1/2}$ where: T = ultimate tensile capacity

Ø = emperically derived constant

f' = compressive strength of concrete

To determine the value of \emptyset , we applied the above written formula to the manufacturer's test data (i.e., ultimate loads and compressive strength of concrete) and back calculated \emptyset . The values for \emptyset , calculated as noted above, are set forth in Table B. While the computed values relate only to the 1 and 1-1/2 inch inserts (the ones of concern), they compare favorably with values computed from other sized inserts.

TABLE B

Richmond Insert Dia. (in) 3/4 7/8 1 1-1/4 1-1/2

Value of Ø .85* .81* .84 .77* .84**

- * Deduced from manufacturer's allowable and a factor of safety of 3.0, not from direct test data, with f' = 2850 psi.
- ** This value is an estimate since the failure mode in the manufacturer's test was rod failure and not concrete failure. However, it is above .79 which is the value calculated assuming concrete failure occurred at rod failure.

This equation is well recognized in industry and extensively used in numerous text books and learned treatises.

Applying the imperically derived values of Ø in equation, and factoring in the range of actual strengths of concrete used at CPSES, the ultimate tensile loads can be calculated. These calculated ultimate tensile loads along with the allowable design loads used at CPSES and the associated safety factor (ultimate load divided by allowable load) are set forth in Table C.

Estimated Ultimate Tensile Loads & Safety Factors For Richmond Inserts

Richmond Size	ø	Allowable Insert Loads Used In Design at CPSES		timated Ultim & (Safety Fac	
			4000 ps1	4500 ps1	5000 ps1
1"	.84	11.5 k	29.8 ^k (2.6)	31.6 ^k (2.7)	33.4 ^k (2.9)
1-1/2"	.84	31.3 ^k	80.9 ^k (2.6)	85.8 ^k (2.7)	90.4 ^k (2.9)

Thus, the estimated minimum safety factors for Richmond inserts in tension which result from the design approach employed at CPSES using actual conditions existing vary in reality between 2.7 to 2.9. (Even had a value of \emptyset = .79 been used, comparable safety factors would result, e.g., 2.7 instead of 2.9.)

It should be noted that out of 912 supports reviewed in Unit 1 and common areas employing Richmond inserts, 865 utilize low strength threaded rods (864 SA-36 and one SA-307 (bolt)). The remaining are high strength threaded rods (45

SA-193, one SA-108, one SA-325). The low strength threaded rods/bolts have lower allowable loads than the allowable loads for the Richmond inserts used in the CPSES design, noted above. Accordingly, while Table C sets forth the allowable loads for the Richmond inserts for pure tension or shear loads, the governing limits on design would not be the allowables for the inserts, but rather the allowable loads of the threaded rods. As a practical matter, however, since inserts and their rods are seldom loaded in pure tension or shear, but are loaded in combined loadings, the governing limit on design will be the interaction ratio for the insert. 5

- Q. On what basis was the shear allowable value established for the 1-1/2 inch insert in the absence of a shear test for that size insert?
- A. The shear value was based on an extrapolation from the existing test data. The test on the 1 inch insert showed that the shear ultimate capacity was approximately equal to the tension ultimate capacity. It also showed that the ultimate shear capacity of the testings anchor studbolt governed rather than the insert's capacity. Therefore, the insert's capacity was actually higher than the shear failure

The interaction ration discussed later in this affidavit for either the insert or the threaded rod is expressed as $\frac{T}{T} + \frac{S}{S} < 1.0$ where T, S, T and S are the tension, a shear, altowable tension and shear in the insert or threaded rod, and n = 4/3 for the insert and 2 for the rod.

load of the test. This prompted the Applicants to set the shear allowable for the insert equal to its pullow. (tensile allowable). Applicants further reduced the shear allowable by multiplying its tension allowable by the ratio of the manufacturer's working shear load (18 kips for 1-1/2 inch insert), to the manufacturer's recommended working tensile load (21.67 kips for 1-1/2 inch insert).

B. Verification Tests

- Q. What tests have been conducted to demonstrate the effect of shear loads on Richmond inserts?
- A. To comply with the directives of the SIT, shear tests were conducted at CPSES on 1-1/2 inch Richmond inserts in March 1983. The test report summarizing those tests is included as Attachment A to this testimony. The salient conclusions of these test, are summarized below.

A total of nine specimens were tested. All utilized 1-1/2 inch type EC-6W inserts in concrete representative of the strength and reinforcement found at CPSES. For the test the concrete strength was approximately 4600 psi. On six specimens a 1 inch thick washer plate was inserted between the shear plate and the insert to represent the washer which is used in pipe hanger installations. Three specimens without washers employed A-490 bolts. Three more specimens with washers also used A-490 bolts, and finally the three remaining specimens (with washers) utilized SA-36 threaded rods.

In no case was the test permitted to go to ultimate failure. Loading application was halted where the load had reached a magnitude considered to be sufficient in comparison with the design load values. (At this point the NRC representative witnessing the test indicated his concurrence).

In spite of the fact that the test did not take the inserts to failure, the results indicated that the performance capabilities of the Richmond inserts in shear exceed the design allowable by a ratio in excess of 3.3 to 1. Because the tests did not go to failure, the actual ratio is higher and the results are conservative.

Moreover, test results for the specimens with and without the l inch thick washer were comparable, indicating that the presence of the washer has little effect on the performance of the threaded connection/bolt or the Richmond insert. If any bending stress is introduced in the bolt as a result of the l inch thick washer, the test results show that it is not significant enough to distinguish the difference. These results justify the shear allowables regarding Richmond inserts used by Applicants in the design of CPSES.

Q. Have other tests been conducted on the Richmond inserts?

A. Yes. As a result of the allegations by CASE that the preceding tests were not sufficient to address combined tension and shear loadings⁶ and the Board's concern with the absence of test data, Applicants proposed a plan⁷ which stated that Applicants would:

"Provide evidence of the capability of Richmond inserts to accept the maximum loads to which they will be subjected in tension, shear and combined tension and shear, with ample margins of safety. The evidence will be generated by a combination of tests and analyses."

To fulfill this plan Applicants performed another series of tests in March and April, 1984. A final report summarizing these tests is included as Attachment B to this testimony.

In summary, these tests were performed to determine the load carrying characteristics of 1-1/2 inch type EC-6W and 1 inch type EC-2W Richmond inserts when subjected to tension only, shear only and combined shear and tension loadings. The strength, deflections and type of deformations produced by these loadings were determined. The tension and shear testing conformed to the requirements of ASTM-E488-81, "Standard Test Methods for Strength of Anchors in Concrete and Masonry Elements." The number of samples of each diameter Richmond insert was in accordance with Section 7 of ASTM-E488-81. However, Applicants are not aware of any

⁶ CASE Proposed Findings of Fact and Conclusions of Law at Section VII and VIII.

⁷ Applicants' Plan at 7.

standará method or test for combined tension and shear. For such tests, tension and shear loads were applied to the test specimen in equal increments, i.e. the tension load was always equal to the shear load. (For a detailed description of the apparatus refer to Attachment B.)

The tests utilized a total of 30 Richmond inserts (fifteen 1-1/2 inch and fifteen 1 inch). To prepare for the tests these inserts and several more spares of both sizes were cast in concrete slabs which utilized the minimum type of surface reinforcement encountered in the field (\$7 grade 60 bars at 10 inches on center in each direction near the surface). The concrete strength was also typical of that encountered in the field, having an average compressive strength in excess of 4900 psi.

To ensure that the tests actually tested the <u>inserts'</u> capacity (and not the capacity of the threaded rods), high strength threaded rods/bolts were utilized in all cases. As previously stated, in field installation it is the threaded rod which most often has the lower allowable load in pure shear or tension. In this regard, in its Proposed Findings at Section VII, CASE has alleged that the wrong allowables for inserts have been used at Comanche Peak. This is not so. The proper allowables for the inserts have been used.

The results of the tests are presented in Attachment B and summarized in Table D, below.

Ultimate Shear, Tensile and Combined
Capacities of Richmond Inserts

Insert Dia.	Tension (T)			Shear (S)		
	Allowable (T _A)	Ultimate (T _U)	FS	Allowable (SA)	Ultimate(S _U)	FS
	11.5 ^k	41.27 ^k	3.59	11.5 ^k	40.28	3.50
1-1/2"	31.3 ^k	101.96 ^k	3.26	27.0 ^k	94.34 ^k	3.49

Combined Shear and Tension

1" 28.35^k (4.15)
1-1/2" 63.47^k (3.68)

+ Utilizes interaction formula
$$(T/T_U)^{4/3} + (S/S_U)^{4/3} = 1$$
.
Factor of Safety in this case is computed from
$$\left(\frac{T}{T_A \times FS}\right)^{4/3} + \left(\frac{S}{S_A \times FS}\right)^{4/3} = 1$$

The test results confirm the judgment of Applicants that (1) shear and tensile ultimate capacities are nearly the same and (2) the actual factors of safety are in excess of 3.0 for shear, tension and combined shear-tension loadings. An important concomitant result of this series of tests is the confirmation of the conservatism of the tension-shear interaction formula utilized for design. This formula, which is suggested by the PCI Design Handbook, Precast and Prestressed Concrete, 1971 at 6-20, states that the interaction between tension and shear goes as the 4/3 power. This formula is verified by the results of these

fall outside the interaction curve, thus providing evidence of the conservatism of the interaction formula.

- Q. What would you conclude from the result of these and prior tests?
- A. We would conclude that the margins of safety for Richmond inserts for loading in shear, tension and combined shear-tension for the conditions expected in the field are in excess of a factor of 3.0.
- Q. In addition to the general concerns raised about testing of Richmond inserts, are there specific concerns about the tests which you wish to address?
- A. Yes. Apparently faced with results of the 1983 shear tests which indicated the significant capacity of the Richmond inserts over design, CASE challenged the validity of the test by alleging that the conditions of the reinforcement in the concrete tests labs did not represent the conditions in the field. As stated in Attachment A, however, the concrete used in the tests was representative of concrete in the plant. Indeed, in Attachment A is the actual test report on the concrete used in the tests. Applicants have conducted a review of a representative sample of test reports of concrete used at CPSES to assure that such concrete is essentially the same as that used in the tests. In addition, Applicants have reviewed NCRs regarding concrete at CPSES to provide additional assurance that the concrete

used in these tests was representative of that used at CPSES. From our review, we conclude that test conditions are representative of conditions at CPSES.

Moreover, to be very conservative, the new tests conducted in March 1984, employed two layers of reinforcement rods rather than 4 layers used in the prior test and at CPSES. As seen in Attachment B, the capacities of the Richmonds were not impaired.

In any event, the difference in reinforcement in the concrete (the concern expressed by CASE) is not significant when compared to other factors. If rebar was a dominant factor, it would be evident from a comparison of the results of the March 1983 tests (using 4 layers of rebar) and the March 1984 tests (using 2 layers of rebar). However, a comparison of those results (including bolt deflections) indicates that the amount of rebar is not a significant factor. See also Tr. 6495-6500 wherein the cognizant Staff witness concurs with this assessment.

III. ABILITY TO RESIST AXIAL TORSION

Q. Are you familiar with the issue regarding the ability to resist axial torsion?

A. Yes. This issue refers to the concern by CASE of the ability of the Richmond assembly (including the threaded rod) to resist "axial" torsion. In the Board's Memorandum and Order of December 28, 1983 at 62, the Board states that this concern is important because

"The Richmond was tested without being connected to a steel member that could induce torsion into the bolt. Consequently, the safety of the Richmond depends in part on the test described in subsection 1., [8] above, and in part on the engineering analysis of the effects of torsion on the bolt."

The Board concurred with CASE's view that the Applicants' manner of computing the tension force in the bolt of the Richmond insert assembly resulting from torsion in the tube steel is incorrect. Id.

- Q. Describe Applicants' method of computing the torsion forces in the bolt.
- A. In computing the torsion force in the bolt of a Richmond insert, the formula T = Fd is used; where T = the torsion applied to the steel tube (see Figures 1 and 2 of Attachment D), F = the tension in the bolt, and d = the distance from the bolt to the force acting on the washer. The Board believed that Applicants were using the distance d as equal to 2/3 of the one half of the width of the washer. See December 28, 1983 Memorandum and Order at 62-66.

This quote refers to the March 1983 test required by the SIT, completed by Applicants, and discussed above.

Applicants, in general, did not use this distance, but instead relied on predeveloped charts which use the distance from the bolt centerline to the centroid of a triangular compressive load distribution, offset from the bolt centerline. When configurations were encountered that are not covered by the predeveloped chart, and for designs performed prior to the development of the charts, Applicants did use the distance questioned by the Board, i.e., 2/3 of the distance between the center of the bolt and the edge of the washer. The distance derived from this calculation is always smaller than that which would be obtained from the predeveloped charts, which is the distance from the centerline of the bolt to the centroid of the triangular compressive load distribution defined between the neutral axis and the edge of the washer. (See Attachment D.)

Since the distances from the charts predeveloped would result in smaller calculated tension in the bolt, we have chosen to focus our discussion on the effects of using this distance (i.e., that obtained from the predeveloped charts) in order to determine whether it accurately reflects the appropriate load distribution.

To illustrate why the Board might be confused as to what distances were used, we will make use of a similar figure (Figure 1 of Attachment D) to that utilized by the Eoard in its Memorandum and Order of December 28 at 77. The major difference between Figure 1 and the Board's figure

(which is included as Figure 2 of Attachment D) is in the meaning of the distance d_2 . This is the distance the Board believes Applicants used in the formula T=Fd. As shown in Figure 2 of Attachment D that distance is equal to 2/3 of the washer half width because it is shown as starting from the center of the bolt.

Applicants generally have used the distance d' from figure 1 of Attachment D, which represents the distance bet seen the centerline of the bolt and the centroid of a triangular compressive stress distribution defined between the location of the neutral axis of bending and the edge of the washer. This axis is not located in the center of the bolt but it is shifted toward the edge of the washer placed in compression by the applied torsion. The location of the neutral axis and the tension in the bolt can be derived by solving the static equilibrium and strain compatibility equations. Such a solution is provided in Attachment D, where it is shown that do is generally greater than do. This clarifies the circumstances which may have confused the Board. The solution for d' provided in Attachment D is correct only if the equation expressing strain compatibility between the concrete and the bolt is valid. While that equation is valid if the problem were truly two dimensional, and is generally employed for solving problems of this kind (see CASE Exhibit 903, Excerpts from Blodgett's Column Base Plates), one cannot say with certainty whether the same form would apply in the three dimensional problem which is present in the field. Because there is no preload (other than snug tightness) of the bolt and hence, no continuity between the tube steel, the bolt, the lower washer and the concrete, the distribution of strains between the bolt and the concrete is a tri-dimensional complex pattern.

- Q. Had Applicants performed any additional analysis to evaluate this complex situation?
- A. Yes. To study this pattern Applicants performed detailed finite element analyses utilizing the STARDYNE computer program. A description of the model and results of the analyses is given in Attachments El and E2. The results of the analyses indicate that the formulas used by Applicants as described above did not precisely model the resulting forces. The formulas used by Applicants resulted in a calculated force that was low for all but six supports by as much as 25 percent. (As noted later in this Affidavit, the finite element analyses refined this calculation and only predicted an 18 percent increase; in addition, because of conservatisms in the methodology and process used, in all cases allowables would not have been exceeded.)
- Q. What did the results of the finite element analyses show?

There are six 4 x 4 x 1/2 tube steel sections loaded primarily in torsion or shear for which this effect would result in a calculated 33 percent increase. This increase has been factored into the interaction formulas in Table 1 (attached) and has been found to be acceptable for the six supports.

- A. The results of the finite element analyses showed the following:
 - a) The transfer of moment (torque) into the couple which results in bolt tension and concrete compression occurs at the tangent point between the tube and the washer. In this respect Mr. Doyle (and the Board) were correct. However, due to the stiffness of the steel, the transfer is along a line and is not spread over an area.
 - b) The compressive force distribution in the concrete is reasonably linear and extends to the edge of the washer. Here, Applicants were right as explained in e) below.
 - c) The quasi-linear force distribution in the concrete, however, is not the same at different locations parallel to, but away from the line drawn from the bolt centerline to the edge of the washer (this is due to tri-dimensional effects) and this is what causes the difference between the original approach used for design and the present results.

The centroid of the triangular distribution existing in the center of the washer (line between center of bolt and edge of washer) coincides vertically with the tangent point of the tube steel and the washer, i.e., the neutral axis adjusts accordingly.

- d) The increase in bolt tension for the worst configuration is less than 25 percent for bolt holes located along the tube steel centerline (see note 9) and this can be calculated by using the expression T=Pd, where d, is the distance between the bolt centerline and the tangent point of the tube steel and the washer.
- e) Applicants ran a sensitivity study and the stiffness of the concrete was varied. For the stiffness existing in the field, the distribution of compressive stresses is essentially linear and extends to the washer as shown in Attachment E2. As the stiffness of the concrete is decreased, the distribution of compressive forces in the concrete becomes non-linear, with the peak of the distribution coinciding vertically with the tangent point between the tube and the washer.
- f) Although not raised as an issue in this case, the finite element model was also executed for the cases in which the bolt holes are offset from the centerline of the tube steel. The offset in the model was equal to the maximum value permitted by the design criteria. This was done to assure ourselves that the largest possible increase in

tension over that computed initially would be determined. Applicants could have used the same method outlined in d) above, i.e., using the lever arm defined as the distance between the bolt centerline and the tangent point of tube steel to washer, to compute the increase in tension on the bolt for offset bolt holes. However, the finite element analyses indicate that this coupling method is not applicable for the bounding eccentricity (which is for 4" x 4" tube steel, 3/4 inch from the center or 1/2 inch from the tangent point of tube steel and washer) which is the worst case that exists in the field.

- g) The finite element analyses discussed in f) above shows that the torsion does not result in a concrete compression/bolt tension couple as discussed above, but rather results in a shear couple at the top and bottom of the bolt which puts the bolt in bending.
- Q. Is there an adverse effect on the safety of the plant from these results?
- A. No. As discussed below, this will result in no adverse effect on the safety of the plant.

Table 1 (attached) lists (Unit 1 and Common) supports using tube steel with Richmond inserts which are safety related and which may be primarily loaded in torsion or shear. This table also lists the existing eccentricities and the loads for the inserts. It is evident that the preponderant number of supports (90%) have tube steel connected to Richmond inserts at the centerline of the tube steel (zero offset) or with small eccentricities. Cases of extreme eccentricities are few (only in about 18 cases out of the 102 cases of 4" x 4" tube steel (mostly loaded in torsion or shear) do eccentricities equal to or exceed 3/8 inch). For the other 53 supports loaded primarily in torsion or shear, only three have offsets equal to or in

excess of one inch (one inch in six and eight inch TS would give a comparable effect as the 3/8 inch in the four inch TS).

For these, the maximum possible underestimation of the tension resulting in the bolt is about 25 percent. (See note 9.) The finite element analyses which will be discussed later actually indicate that the maximum experienced increase is only 18 percent. This 25 percent corresponds to the difference between the proper lever arm, i.e., that between the bolt centerline and the tangent point of tube steel to washer, and that used in design for the most common 4" x 4" tube steel (thickness = 3/8 inch). Other tube steel dimensions will have lower differences. (See note 9.) The 25% increase (and the 33% increase for the 4"x 4" x 1/2" tube steel cases) can be accommodated by the supports.

In the process of performing the finite element analyses, described in Attachment E, Applicants noted that when it is assumed that no clearance exists between the tube steel and the bolt, a shear couple is created which places the bolt in bending. The effect becomes pronounced when the bolt holes are offset to their largest values. The prior manual or chart methods of analyses cannot account for the bending effect. To investigate the possible adverse effects on the connections Applicants developed a screening

criterion, based on a very conservative analysis, by which we could judge which particular supports require closer scrutiny.

This criterion requires that any connection where either the insert interaction exceeds unity or the bolt interaction equation exceeds 1.75 must be listed as a candidate for further evaluation. The factor of 1.75 for the bolt derives from two factors, each having a value of 1.33, which represent, respectively, the difference between the bolt bending stresses predicted by finite element analyses and those predicted by simple flexure manual calculations (the latter are 33 percent higher, as indicated in Attachment E3), and the difference between values of .75 F, (the allowable bending stress) and F, (where F, is yield strength of bolt material). For establishment of the criterion, Applicants allow the outer fiber stresses of the bolt to reach yield, because the manual method of analysis employed to compute such stresses has been shown by the tests discussed in Attachment F to be extremely conservative.

The factors of safety inherent in the methods of calculation employed to establish the interaction ratios needed for the criterion are shown in Table 1 of Attachment F (method D) and are shown to be in excess of 10. The method of computation of the interactions is summarized below.

A portion of the torsional moment is applied to the bolt as a bending moment, which accounts for the internally created shear couple. Depending on the offset of the bolt hole, different fractions of this moment are inputted as direct bending moment of the bolt. For any offset exceeding 1/4 inch, all the moment is inputted as bending moment of the bolt. Even with zero offset, 38.4 percent of the external moment for 1-1/2 inch bolt (17 percent for the 1 inch bolt) is applied to the bolt as a bending moment.

The moment in the bolt induced by the shear is determined by multiplying the shear value by the distance from the center of the tube steel to the concrete and multiplying this times 0.58 for 1-1/2 inch bolts with no offset (or 0.72 for 1 inch bolts with no offset, or 1.0 for bolts with offset). On Any fraction of the moments not inputted into the bolt as bending is coupled out into bolt tension as described for the traditional method. The Board

¹⁰ The fractions of the moments (where these fractions are 0.58, 0.72 and 1.0 for 1-1/2 inches with no offset, 1 inch with no offset and 1 inch with offset, respectively) that are assumed to go into inding are extrapolated from the recent worst case shear inite element analyses conducted on a single size tube steel ("TS") (4" x 4" x 3/8") and prior analyses (also conducted on 4" x 4" TS) performed in September of 1982. (SIT Report at 21.) Since none of these analyses were conducted at intermediate offsets, a linear distribution of the fraction of external moment going into the bolt as bending is assumed from zero offset to an offset of 1/4 inch. Above 1/4 inch offset all the external moment is assumed to go into bending the bolt. Also, for any offset all of the bending due to shear is assumed to go into the bolt.

should recall that in the traditional method of analyses discussed previously, all of this moment would be coupled out as tension in the bolt. Any external pull is added to this tension to give the total tension. The resulting tension, applied shear, and bolt bending are used in the following bolt interaction equation:

$$(T/T_A)^2 + (s/s_A)^2 + (M_b/M_{ba}) = bolt interaction ratio$$

where ${\rm M}_{\rm ba}$ is the allowable bolt bending moment as computed from ${\rm M}_{\rm ba}{\rm c/I} = 0.75~{\rm F}_{\rm y}$, ${\rm T}_{\rm A}$ is the allowable bolt tension and ${\rm S}_{\rm A}$ is the allowable bolt shear. The tension (T) equals the applied external tension plus any coupled-out tension resulting from torsion. The shear (S) is the applied external shear, and ${\rm M}_{\rm b}$ (the applied bolt bending moment), has been defined above. The bending moment in the bolt is converted to a couple within the bolt (moment arm = effective diameter of the bolt).

The total pull of the insert, $T_{\rm IP}$, defined as the equivalent total axial load, is calculated by adding the tension component of the bolt internal couple to the tension, T, calculated above. This total insert pull and the applied shear are used in the insert interaction equation, noted below,

$$\frac{T}{T_{AI}}^{IP} + \frac{S}{S_{AI}}^{4/3} = insert interaction$$

where T_{AI} is the allowable insert tension and S_{AI} is the allowable insert shear, T_{IP} is the total insert pull and S is the shear on the insert.

The manner in which these interaction ratios are computed is based on very conservative assumptions (see e.g., note 10), which were not borne out by the testing noted in Attachment F (e.g., the tests indicate that larger offsets are needed for these limiting conditions to be valid and that even at the largest offset not all of the moment goes into bending). For the larger tube steel sizes (i.e., greater than 4" x 4") the conservatism is compounded since the same percentages were used whereas the effect of the offset would be progressively smaller.

Table 1 (attached) summarizes the results of the evaluation of the interaction ratios for the safety related supports which can experience loads primarily in torsion.

From Table 1 (attached) there are a total of 12 supports which exceeded the interaction ratio. These mostly fall in the following categories:

(a) tube steel connections with relatively large offsets, and

(b) tube steel connections with smaller or zero offsets which employ 1 inch bolts, which by virtue of the small section modulus of the bolt are less capable of withstanding bending loads.

Although Applicants are concerned with the conservatively calculated bending stresses in the bolts, from the results of testing noted below, there is no safety concern with these connections.

Of the tests reported in Attachment F, the most adverse test is the torsional test of the 4" x 4" x 3/8" TS insert with the 3/4 inch offset which indicated failure (or near failure) at approximately 10,600 lbs (applied 2 inches above the top of the tube steel). The configuration of this test is designed to encompass many of the supports listed in Table 1 (attached). If the 4" x 4" x 3/8" connection with a 1-1/2 11ch bolt having the highest torsion and shear is examined against the test results the following is noted. This support, CT-1-053-408-C62R, is computed to exceed the interaction ratio criterion when subject to a shear load of 2.479 kips and a torsion of 9.249 in-kips, with no offset. The test conducted for the 4" x 4" x 3/8" tube steel with a 3/4 inch offset (which is worse than that of the related support) loads the connection in torsion and shear. When the shear equals 3 kips, the corresponding torsion is 21 in-kips. At this loading condition, the measured deflection of the assembly is 0.05 inches, which is 6 percent of the

ultimate deflection. The factor of safety to failure for the support (load = 2.479 kips shear and 9.249 in-kips torsion) is greater than 4 based on the test results. Thus, even though the interaction ratio criterion indicates that the worst case support, CT-1-053-408-C62R, may be suspect, the test shows that there is no safety concern, and that an adequate margin of safety exists.

Applicants recognize that the criterion and method employed and determine whether the bolts can accept the loads in these instances is not covered by the Code. The Code does not provide for such eventuality, as it assumes bolts to be loaded in shear and tension only. The bolts can indeed accept the shear loads, but tension has no real meaning when greatly offset holes are present. As is evident from Attachment F and also the finite element analysis of Attachment E, the shear couple generated in such instances gives rise to a combination of bending, tension and shear of the bolt, for which the Code makes no provision. The tests support the conservatism of the chosen approach. (It should also be noted that from the test results shown in Attachment F, one can verify that tube steel deformations for the applied loads are low.)

IV. Method Used To Analyze Connections 11

- Q. Have you reviewed the issue regarding methods used to analyze connections?
- A. Yes. In Section VIII of CASE's Findings, Messrs. Walsh and Doyle expressed concerns over the methods used to analyze the connections of the bolts, tube steel and Richmond inserts. Specifically, this concern focuses on the acceptability of release of all moments except for the torsional moment (M).

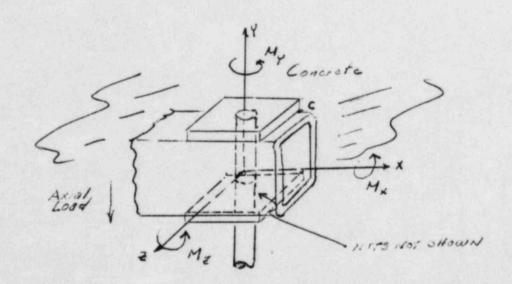
CASE agrees that the moment in the tube (My) about the axis of the bolt cannot develop, but they state that the other moment (Mz) (which would tend to produce prying action, if any), should either be considered whenever the moment which produces torsion (Mx) is considered, or both Mx and Mz should be released. CASE states further at VIII-6 that "the ability to rotate about the local Z axis is inhibited; therefore, prying (moment coupling) exists."

(Refer to Figure 1 for an explanation of the coordinates and moments.)

In the area, CASE's concern regarding the method selected by Applicants to react the shears is addressed in the preceding discussion of the ability to resist axial torsion.

FIGURE 1

EXPLANATION OF COORDINATES & MOMENTS



To examine the validity of this concern we have utilized a finite element analysis which employs the same model and method as the analyses described in Attachments El and E2, and which examines the behavior of the joints under the combined influence of axial (parallel to the insert bolt, Mz) and torsional loads or purely axial load. purely torsional load was addressed separately via another finite element analysis, referred to previously. Clearly for single tubes loaded in torsion, the restraint of torsional moment is required for stability. Similarly, for single tubes loaded torsionally and axially, the axial displacement resulting from the maximum permissible axial load in the tube is insufficient to prevent the torsion constraint as discussed below. Moreover, the single tubes are all lightly loaded, further pointing to the correctness of modelling the torsional moment constraint. The resistance of the attachment assembly under pure torsional loading was demonstrated to develop bearing between the tube and upper shim plate solely along the line of tangency at the corner of the tube. The couple between the bearing area and the bolt tension equals the applied torsional moment; therefore, the prying action in the bolt can be calculated directly.

due to bolt elongation (along the Y direction) is sufficient to cause loss of contact with the washer. Thus, there is no prying action. For pure axial loads, i.e. loads applied to the tube steel between Richmond inserts in the y direction, there is no prying action and their release of the moment about the Z axis is the correct way to model the joint.

A parametric study of the loading was performed to analyze the effect of bending moment M_Z on the prying action which occurs due to the torsional load. For the study, a 4 x 4 x 3/8 inch tube with 1-1/2 inch diameter inserts located 20 inches on center was analyzed.

The bending moment is introduced by the addition of an axial load at the center of the attachment assembly.

Two parameters were analyzed:

- Variable applied bending load with constant torsional load.
- b. Variable torsional load with constant bending load. Analyses were performed for the load cases shown below in Table E. Additional data presented include the fixed end moment ("PEM") calculated for the applied pull load had the connection been modelled as fixed with respect to the M_Z in STRUDL, and the ratio of the FEM to the applied torsional load ("FEM/Torsion").

- 35 -

TABLE E

LOADING NUMBER	TORSIONAL LOAD (in-lbs.)	AXIAL LOAD (lbs.)	FEM (in-lbs.)	FEM/ Torsion
1	4000	2000	5,000	1.25
2	4000	8000	25,000	5.0
3	4000	20000	50,000	12.5
4	4000	40000	100,000	25.0
5	1000	40000	100,000	100.0
6	0	40000	0	0

Each load case was analyzed to identify the mode of resistance of the assembly. Results for the first five analyses showed the area of bearing between the structural tube and the top shim plate to be limited to the line along the tangent point of the tube corner. Any bending resistance is developed by the eccentricity due to translation of the torsional resistance toward the end of the tube. The sixth analysis showed that no bending resistance was developed in the absence of a torsional moment.

Table F summarizes the results for each load case.

Information tabulated includes the following items:

- a. Loading-torsion (in-lbs.); pull (lbs.)
- b. Expected bolt reaction neglecting bending in the bolt proper $(lbs.)^{12}$

¹² In computing the bolt reaction, the axial load was added to the tension computed from the torsion by the point-oftangency method.

- c. Bolt reaction from analysis (lbs.)
- d. Maximum possible bending resistance with torsional loading governing prying action (in-lbs.)
- e. Bending resistance from analysis (in-lbs.)

TABLE F

Loading No.	Loading Torsion	Pull	Expected Bolt Load	Actual Boit Reaction	Max Bending ¹³ Resistance	Actual Bending Resistance
1	4000	2000	2600	2600	3200	1618
2	4000	8000	5600	5600	3200	2684
3	4000	20000	11600	11500	3200	2966
4	4000	40000	21600	21400	3200	2886
5	1000	40000	20400	20300	800	600
6	0	40000	20000	20001	0	0

The flexibility of the connection under bending is due to the elongation of the bolt from the tensile loads.

Loading No. 6 demonstrates that there is no bearing between the tube and the washer plate if torsion is not present.

This moment resistance is established by assuming (from finite element analysis) that the reaction to the combined torsica and axial load (which results in the M_Z moment) occurs at the intersection of the line of tangency and the edge of the washer (point C of Figure 1). The distance between that point and the center of the bolt is 2 inches in the x direction (M_Z lever arm). For example, the reaction due to the applied torsion at that point is 1600 lbs. for a 4000 in-lb. torsion (this is computed from $\frac{4000}{2(1.25)}$). Thus, the resistance to the

moment about the z axis due to the torsion reaction for this case is 3200 in-lbs. No increase in bolt tension would occur until this resistance is exceeded as a result of the pull. However, when the actual bending resistance (obtained from the finite element analyses which considered both torsion and bending (M_)) is compared to the max-bending resistance due to pure torsion, it is seen that the actual value is always lower, indicating no prying action from the bending.

Based on the results of this study, it is evident that any additional bolt tension need only be considered when torsional loads are present. The increased tension can be calculated directly from the ratio of the torsion and distance from the bolt centerline to the tangent line of the corner of the tube. It is also evident that modelling the joint with the M_z moment released is a more correct manner than modelling it as fixed because of the low bending resistance of the joint. Applicants recognize, and calculations demonstrate, that modelling of the joints as pinned instead of fixed would result in stresses and deflections of the member steel tubes which are higher than those which would be calculated on the basis of fixed connections. On the other hand, fixity of the connection results in higher loads on the inserts. Analyses indicate that the percentage increase in member loads resulting from releasing all moments is not nearly as large as the decrease in load of the insert. Design of the connection with the assumption of a M moment constraint produces conservative loads for the Richmond inserts, which are generally the limiting factors, while producing loads on member steel which are minimally unconservative. Table G, below, shows the M moment carrying capacity of the lightest tube steel section for large bore piping and of the 1-1/2 inch insert connection based on the equation

M_{max} Tube Steel = .6 F_y x Section Modulus; Insert M_{max} = Allowable Tension x Lever Arm from bolt centerline to tangency point.

TABLE G

TS Size	Section Modulus	Tube Steel M	Insert M max
		(in-kips) max-	(in-kips)
4x4x1/4	4.11	92.22	42.16
6x6x1/4	10.1	226.64	84.33
8x8x1/4	18.8	421.87	112.44
10x10x1/4	30.1	675.44	140.55

This shows that the insert is the limiting factor by at least a factor of 2. The difference in the bending moment between a member with pinned ends and a member with fixed ends is less than 2. Therefore, if a support was modelled with Mz fixed, releasing Mz would lower the insert loads, increase the tube steel bending moment, but not overstress the tube steel.

prior to beginning the as-built program, NPSI began analyzing the joints as pinned. If the designer was not sure whether the pinned model was correct he would check if there was sufficient elongation in the bolt to allow the rotation of the tube steel. The use of the pinned assumption is normal structural design practice. In fact, the 8th Ed. AISC Specification, paragraph 1.15.4, states that inelastic action in the connection is permitted to accommodate end rotations.

PSE leaves it to the designers' judgment to decide whether the moment should be released and, therefore, has not always reanalyzed the joints during the as-built program as pinned. PSE has in some cases still retained constraint on the M_Z moment. Even though the finite element analyses indicate that this is an appropriate modelling assumption, we would like to place in perspective the effect of this assumption on the steel member stresses.

Applicants have reanalyzed several support configurations selected at random assuming that all moments would be released. Table 2 (attached) provides a comparison between the maximum stresses and deflections of the members calculated with and without the constrained moment. Also shown in this table are the margins to allowable loads which exist. As can be readily seen, adequate margins exist, even with the fully released moments. As a final point, the effect of modelling on the support stiffness should also be addressed.

case contends that the difference in modelling can result in substantially different stiffnesses, and hence, invalidate the assumption of generic stiffness being applicable to the piping analysis. Applicants have addressed the issue of generic versus actual stiffnesses under a separate affidavit, see Applicants' Motion for Summary Disposition Regarding Use of Generic Stiffnesses Instead of Actual Stiffnesses In Piping Analysis, filed on

May 21, 1984. However, it is important to state here that significant effects from differences in stiffnesses do not occur unless the differences between adjacent supports or groups of supports are fairly large. As seen from Table 2 (attached), the difference in stiffness is not great enough to have a significant impact on the piping analyses.

V. Bending Moments

- Q. Are you familiar with the issue of bending moments?
- A. Yes. In section VIII of CASE's Findings, CASE is concerned with allegedly high bending moments in the bolt resulting from the imposition of a shear force on the bolt offset from the concrete surface by the use of a one-inch washer between the concrete and the support steel.

Bending of the bolt is not considered by the ASME Code, because in conventional bolt connections, bending is not significant. In reality, however, bending can occur. This problem was addressed by the SIT14 which had indicated that Applicants' preliminary calculations showed the bending moments to be insignificant in all but one of 60 cases reviewed. The NRC in the same report requested that the total stress (including the bending stress) in the bolts should be evaluated to assure that the value for allowable stress has not been exceeded.

¹⁴ SIT Report at 21.

There are two possible ways for the joint to react to the bending moment and, therefore, two ways to analyze them. One way is to compute the increased tension in the bolt by the same method as that used for the applied torsion moment (only now using the lever arm from the center of the bolt to the point of tangercy). This is not an entirely correct manner because the bending moment would also be reacted by a couple internal to the bolt. This approach would then be an approximate approach, perhaps non-conservative, which would resolve the bending moment into an increased tension to be included in the shear-tension interaction formula of the Code. The second, conservative approach is to compute the bending stresses from the Mc/I formula or finite element analyses, then add the bending stress ratio-to-the-allowable (conservatively assumed as 0.75 F_v where F_v is the yield stress) to the Code interaction formula in linear fashion. As discussed previously in Section III (Ability To Resist Axial Torsion), Applicants have used the latter approach in evaluating the supports of Table 1 (attached) which are highly loaded in shear (which include those among the 60 supports mentioned by the SIT). The results of these analyses are set forth in Table 1 (attached), and, as discussed in Section III, reflect that due to the conservatism of the calculational methodology bending does not present a safety concern with these connections.

The results of tests reported in Attachment F reinforce Applicants conclusion in this regard, (i.e., that deflection of the supports at the design loads are very small regardless of whether the load is applied torsionally or as a shear, and that ample margin exists.

It should be further stated that about fifty percent of the bending moment in the bolt (from shear loading) is contributed by the shear at the tube steel flange next to the concrete. The shear tests conducted in March 1983 without tube steel (but with the washer) would also have contributed a bending moment to the bolt, and hence, those results provide corroboration that there is ample margin against failure.

VI. Sharing of Shear Loads

- Q. Are you familiar with the issue regarding sharing of shear loads?
- A. Yes. CASE's allegations in this regard are concerned with the sharing of the shear load among all of the bolts in a particular support. CASE alleges that only half or fewer of the bolts would accept the shear and would exceed allowable values before the remainder take up the load because of the presence of oversized bolt holes. We believe that their concern is not valid. Since this concern is common to all connections, not just to Richmond inserts, we have chosen to discuss it more fully in a separate Affidavit and Notion for

Summary Disposition Regarding the Effects of Gaps on Structural Behavior Under Seismic Loading Conditions filed in this proceeding on May 18, 1984.

VI. Additional Matters

- Q. Does this complete your testimony on matters relating to "Richmond inserts?
- A. Almost. As a final point, we would like to address the concern (raised on VIII-11 of CASE findings) that Applicants failed to consider the A-307 bolt in their calculations submitted as Applicants' Exhibit 142D.

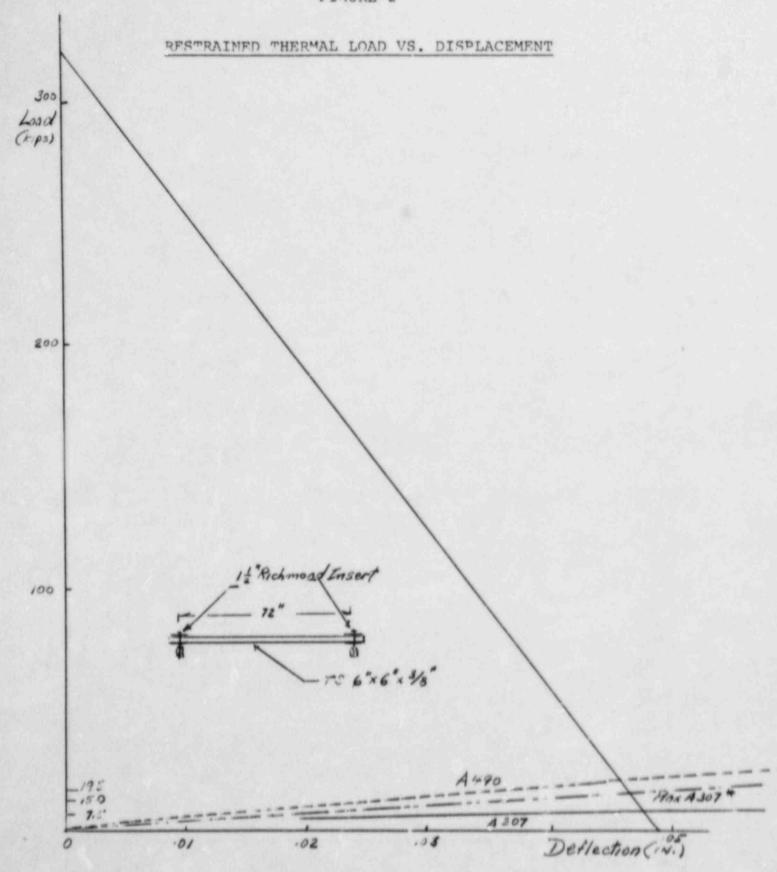
Applicants did not fail to consider the A-307 bolt; they purposely did not include the strength of the A-307 (A-36) bolt because the purpose of the analysis was to demonstrate that even the stiffest anchorage possible would considerably relieve the thermal expansion stresses resulting from LOCA and that the resultant load on the anchor would be considerably smaller than that computed for a fully restrained structural member. This was the purpose of Applicants' Exhibit 142D. It should be clear to everyone that the highest load on the anchorage system results from assuming the least flexible member of that system. If a high strength bolt were used for the Richmond insert, the least flexible member may or may not be the insert. However, both the test data obtained from the manufacturer and that obtained by Applicants (Attachment B to this

testimony) certainly indicate that the failure occurs in the bolt rather than the insert, pointing to the latter as being the stiffer and stronger member of the anchorage system.

Thus, use of test data acquired via high strength bolts is appropriate if one wishes to determine the maximum load on the Richmond anchor, so that this load can be compared against the insert allowable. This, of course, was not the purpose of Applicants' Exhibit 142D.

Nevertheless, just to make the obvious point,
Applicants recognized that A-36 rods are more flexible than
high strength bolts, and that they have lower allowable
values than the Richmond inserts, i.e., 17 kips instead of
25 kips. Applicants, however, also recognized that the
thermal expansion load that would occur had an A-36 rod been
used, is lower than that calculated for the high strength
bolt. This load would then be the one that should be
compared against the allowable load for the A-307 bolt. To
put this concern in perspective, the thermal expansion load
that would have resulted from the use of an A-307 bolt is
seen from Figure 2. Also shown in this figure is the load
computed for a high strength bolt. Figure 2 is developed
using the March 1983 and 1984 test data (Attachment B) using
the methodology employed in Applicants' Exhibit 142D.

FIGURE 2



The load resulting from the thermal expansion for the stiffest connection employing A-36 threaded rod is 5.0 kips. This load is below the allowable 17 kips for the A-307 bolt. When the maximum allowable mechanical load (17.7 kips) is added to the thermal load (per procedure of Exhibit 142D), the resulting deflection would be 0.4 inch. The ultimate deflection is about .95. Thus, there is a margin of safety of 2.4. The ultimate load is approximately 61 kips; hence, the safety factor on a load base is also 2.7. To finish this argument, it is appropriate to again place the purpose of Applicants' Exhibit 142D in perspective. Its purpose was to demonstrate the self-limiting nature of the thermal expansion load and why it need not be considered since anchorage slippages are minute with respect to the ultimate slippage capacity.

Robert C. Totti

Sworn to before me this 1st day of June 1984.

Melena Mellic Notary Public

My Commission Expures Tabeutry 14, 1988

John C. Finneran, Jr.

Sworn to before me this 1st day of June 1984.

Religio Miles

to Commission Cupies February 14, 1986

R. Peter Deubler

Sworn to before me this 1st day of June 1984.

Notary Public

The State of Torbus Toberony II, 1999

LIST OF ATTACHMENTS

I. Documents

- A. Test Report Shear Tests of Richmond Inserts March 1983
- B. Test Report Shear and Tension Tests of Richmond Inserts -April 1984
- C. Combined Shear and Tension Test Results Summary -Shear/Tension Interaction Curve
- D. Original Design Approach Three Equation Method
- E. Finite Element Analyses
 - El Finite Element Model
 - E2 Finite Element Results
 - E3 Finite Element Model and Results for Bolt Bending
- F. Test Results for Inserts and Tube Steel Loaded in Shear and Torsion
- G. Qualification Statement of Day Fotor Doubler

II. Tables

- 1. Richmond Inserts Subject to Torsion
- 2. Tube Steel and Richmond Inserts Comparison of Results Obtained With STRUDL With and Without Releasing M_Z .

ATTACHMENT A

TEST REPORT

SHEAR TESTS

ON

RICHMOND 1 1/2-INCH TYPE EC-6W INSERTS

MARCH 30, 1983

Prepared by

J.C. Hilbrith, P.E.

Civil Engineer

Approved by

R.M. Kissinger

Project Civil Engineer

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 - APPENDIX 1 - DRAWING NO. FSC-00464, SHT. 1
 - CONCRETE COMPRESSIVE TEST REPORT
 - APPENDIX 2 - TEST DATA SHEETS
 - APPENDIX 3 - LOAD-DEFLECTION CURVES

TEST REPORT

SHEAR TESTS

ON

RICHMOND 1 1/2-INCH TYPE EC-6W INSERTS

1.0 REFERENCES

1-A CP-EP-13.0 Test Control

1-B CP-EI-13.0-8 1 1/2" Richmond Insert Shear Tests

2.0 GENERAL

2.1 PURPOSE AND SCOPE

These tests were performed to determine the characteristics of Richmond 1 1/2-Inch Type EC-6W Inserts when installed in concrete representative of that used in the power block structures at CPSES and subjected to shear-type loading. The strength, deflections, and type of deformations produced by this loading were the qualities to be determined. This series of tests employed only 1 1/2"-Inch Type EC-6W Inserts subjected to shear loads.

2.2 RESPONSIBILITY

The tests were performed under the direction of the CP Project Civil Engineer. Witnesses to the tests were: A Nuclear Regulatory Commission (NRC) Representative from the Arlington, Texas Regional Office, the NRC Inspector stationed at CPSES, a TUSI site Quality Assurance representative, and other site engineering personnel.

2.3 TEST APPARATUS

The arrangement and details of the test apparatus are shown on Drawing No. FSC-00464, Sheet 1, included in Appendix 1 to this report. The insert specimens tested were taken at random from the Constructor's stock on site and were; therefore, representative of those installed in the plant structures. They were placed in a thick concrete slab cast specifically for these tests and which was composed of materials and reinforcement similar to those elements of the plant buildings. This is "4000-pound concrete" (28-day strength). The laboratory test report on the concrete of which this slab is composed is included here in Appendix 1.

An apparatus for applying shear loads to the specimens was designed and built on site. This facility employed a 60-ton capacity manually operated hydraulic ram whose thrust against a crosshead was transmitted by tension rods to a 1 1/2-inch thick shear plate bolted to the insert specimen. Base reaction of the ram was transmitted through a structural steel grillage to the outer face of the concrete slab. Ram thrust was determined by multiplying the fluid pressure (PSI), as indicated by a gauge on the pump, by a number equal to the ram piston area in square inches. Deflections were measured by a dial indicator mounted on a remotely anchored bracket and with its springloaded probe in contact with the specimen bolt head or bottom nut where threaded rods were used. These instruments bore valid stickers showing them to be currently in calibration.

3.0 PROCEDURE

In performance of the tests, inserts were cleaned of concrete mortar and other trash that would affect bolt thread engagement. The shear plate was attached to the specimen insert by a suitable length bolt or threaded rod of type shown on the test data sheets, Appendix 2. A new and different bolt was used for each insert. These fasteners were tighteded "snug tight". On three specimens the shear plate was attached in direct contact with the top of the insert. On six other specmens a 1-inch thick plate was inserted between the shear plate and the insert, representing the "washer" used frequently at this location in pipe hanger installation. Shear loads were applied by the ram by operation of the manual pump. As the load increased from zero (o), indications of fluid pressure (later converted to load) and bolt head deflection were read at regular intervals . These intervals were at 400 PSI on the pressure gauge, corresponding to 5300 pounds thrust. Load application on each specimen was halted before failure occured and when the load had reached a size considered to be sufficient in comparison with the design load values. At this point in each test, the NRC Representative indicated his concurrence with this consideration. After this, the load was removed, the apparatus detached, and observation was made of the condition of the specimen.

4.0 RESULTS

As can be seen on the test data sheets, the maximum load applied to specimens on which ASTM A490 bolts were used ranged from 88,110 lb. to 95,400 lb.. The bolts could be seen, after removal from the insert, to be slightly bent. By measuring the distance of the bolt tip from a line perpendicular to the bolt head these deflections were approximately as follows:

Fastener Type	Specimen No.	Bolt Length	Deflection of Tip
A-490	1	4 1/2-in.	0.0 in.
A-490	2	5 1/2 in.	0.05 in.
A-490	3	5 1/2 in.	0.10 in.
A-490	4	4 1/2 in.	0.05 in.
A-490	5	5 1/2 in.	0.10 fm.
A-490	6	4 1/2 in.	0.0 in.

Other than these deformations, no bolt showed signs of incipient failure.

Loading of the three specimens employing a double-nutted SA-36 threaded rod for attaching the shear plate and including the 1-inch washer plate produced a reverse curve in the threaded rod. The offset between the approximately parallel ends of each rod was approximately as follows:

50	pecimen No.	Offs	et
	7	0.4	in.
	8	.4	in.
	9	.4	in.

The fact that the end portions of rods were not truly parallel accounts for the difference in deflection measured at the bottom nut on the rods. Although these deflections were experienced, there was no sign of imminent failure of either the threaded rod, the insert, or the concrete.

There was small spalling of concrete around the top of some inserts. This allowed the top of insert to deflect laterally and in the case of Specimen No. 1 to deform to a small extent. However, in no part of any test specimen did breakage or complete failure appear to be imminent. In each case at the time operation of the hydraulic pump was nalted, the applied load was increasing, showing that neither the insert nor fastener had reached its maximum load carrying capability.

The factor of safety for each specimen based on these maximum applied loads is snown in the following table.

FACTORS OF SAFETY

BASED ON

MAXIMUM APPLIED LOAD

Fastener	Specimen Number	Maximum Applied Shear Load (Kips)	Factor of Safety F.S. = Max. Applied Load Design Allowable Load
	1	88.1 *	88.1/26.51 = 9.32
A-490 Bolt W/ I" Shim #	3	90.1	90.1/26.51 = 3.40
7 1 3/11/11 /-	5	95.4	95.4/26.51 = 3.60
	2	95.4	95.4/26.51 = 3.60
4-490 Bolt Wo 1" Shim #	4	95.4	9.5.4/26.51 = 3.60
	6	90.1	90.1/26.51 = 3.40
SA-36 Threaded Rod W/ I" Shim P	7	58.3	58.3/7.67 = 3.30
	8	63.6	63.6/17.67 = 3.60
	9	63.6	63.6/17.67 = 3.60

^{*} Load halted due to dial indicator for deflection having reached its limit of travel.

5.0 CONCLUSION

These test results show that the performance capabilities of the Richmond Insert in shear exceed the design allowable by a ratio of more than 3 to 1. Thus, a minimum factor of safety of 3 is indicated. The test results for the specimens with the 1" thick washer are comparable to the test results for the specimens without the washer. This indicates that the presence of the washer had little effect on the performance of the bolt or the Richmond Insert. If additional bending stresses are introduced into the bolt as a result of the presence of the 1" thick washer, the test results show that it is not significant enough to distinguish the difference.

Based on this test, the design allowables for shear loading are acceptable for use without further investigation or additional calculations.

APPENDIX 1

TUGCO

LAS SUPETVICOR

TUGCO

LAB SUPSULISOR

APPENDIX 2

TEST DATA SHEETS

RICHMOND 1-1/2-INCH TYPE EC-6W INSERTS SHEAR TESTS REFERENCE: CP-EI-13.0-8

	A-490		SHIM PL. W/O SHIM PL.
DEFLECTION (IN.)	GAUGE PRESSURE (P.S.I.)	JACK* THRUST (LBS).	NOTES - FAILURE MODE
0-190	200		
42	800		
-23	1200		
1.023	400	5,300	
0.04	800	10,600	
.655	1200	15,900	
.083	1600	21,200	
,105	2000	26.500	
.138	2400	31,800	
.168	2800	37,100	
,200	3200	42,400	
,230	3600	47,700	
,270	4000	53,000	位 多数 化多位性 化二氯甲基甲基甲基甲基甲基甲基甲基甲基甲基甲基甲基甲基甲基甲基甲基甲基甲基甲基甲基
.206	4400	58300	Street Yield - Jack bad How
. 360	4800	63,600	Sunta Yield - Jack bod How due prolesty to pump be
.452	5200	68,900	
.530	5400	74,200	
.413	6000	79,500	
.377	6400	84,800	2 1.
1.000	6600	88,110	Frant int over-telig wie

JACK THRUST (LBS) = GAUGE PRESSURE (P.S.I.) TIMES 13.25

JACK: EQUIPMENT NUMBER ACH 606

PRESSURE GAUGE: M&TE NUMBER 1821

DIAL GAUGE: M&TE NUMBER 2094

PERFORMED BY:

DUE DATE: 9 June 83

DUE DATE: 20 june 83

WITNESSED BY:

DATE DATE

Vistor Helden 3-22-93

RICHMOND 1-1/2-INCH TYPE EC-6W INSERTS SHEAR TESTS REFERENCE: CP-EI-13.0-8

EFLECTION (IN.)	GAUGE PRESSURE (P.S.I.)	JACK* THRUST (LBS).	NOTES - FAILURE MODE
0.010	400	5,300	
.028	800	10,600	
.062	1200	15,900	
.094	1600	21,200	
.130	2000	26,500	
,172	2400	31,800	
.214	2800	37,100	
.254	3200	42,400	
.285	3600	47,700	
.306	4000	53,000	
,326	4400	58,300	
.348	4800	63,600	The state of the s
.371	5200	68,900	
.400	5600	74,200	The state of the s
.434	6000	79.500	THE REPORT OF THE PARTY OF THE
.472	6400	84,800	
.513	6800	90,100	
560	7200	95,400	Concrete L'ailei - come type
			in leder - Prying action.
			Sial :- arginoc '7" dian.

JACK THRUST (LBS) = GAUGE PRESSURE (P.S.I.) TIMES 13.25

JACK: EQUIPMENT NUMBER RCH 606

PRESSURE GAUGE: M&TE NUMBER 1821 DUE DATE: 9 20 23

DIAL GAUGE: M&TE NUMBER 2094 DUE DATE: 20 20 23

PERFORMED BY: WITNESSED BY:

DATE STANZETTE - 22 -80

DA REPRESENTATIVE DATE

RICHMOND 1-1/2-INCH TYPE EC-6W INSERTS SHEAR TESTS

REFERENCE: CP-EI-13.0-8

OLT SPEC:	A-470	W/	/SHIM PL. W/O SHIM PL.
DEFLECTION (IN.)	GAUGE PRESSURE (P.S.I.)	JACK* THRUST (LBS).	NOTES - FAILURE MODE
0.018	400	5,300	
0,053	800	10,600	
0,080	1200	15,900	
0.130	1600	21,200	
.145	2000	26,500	
.175	2400	31,800	
,207	2800	37,100	
.248	3200	42,400	
. 304	3400	47.700	
. 365	4000	53,000	
.417	4400	58,300	
.463	4800	63,600	
.508	5200	68,900	HE WEST CONTROL OF THE SECONDARY
.539	5600	74,200	Concrete state & spall
.612	6000	79,500	
668	6400	84,800	
.725	6800	90,100	
	No. of the		Concrete spalled around up
			part of insert permitting extens
			deflection of insert.

*JACK THRUST EQUAL SHEAR LOAD ON INSERT.

JACK THRUST (LBS) = GAUGE PRESSURE (P.S.I.) TIMES /3.25

JACK: EQUIPMENT NUMBER RCH 606

PRESSURE GAUGE: M&TE NUMBER 1821 DUE DATE: 9 Line 83

DIAL GAUGE: M&TE NUMBER 2094

PERFORMED BY:

DUE DATE: 20 june 83

WITNESSED BY:

Cae Villett. 3.22-83

RICHMOND 1-1/2-INCH TYPE EC-6W INSERTS SHEAR TESTS REFERENCE: CP-EI-13.0-8

SPECIMEN NUMBER:		DATE 22 7/ Lun 83	
BOLT SPEC: 4 - 490	W/SHIM PL.	W/O SHIM PL.	_

DEFLECTION (IN.)	GAUGE PRESSURE (P.S.I.)	JACK* THRUST (LBS).	NOTES - FAILURE MODE
0.009	400	5,300	
.019	800	10,600	
.043	1200	15,900	
.070	1600	21,200	
.100	2.000	26500	
.132	2400	31,800	
.165	2800	37 100	
.198	3200	42,400	
.249	3600	47700	
.308	4000	53,000	
.380	4400	58,300	
.448	4800	65,800	
.5-//	5200	68,900	
-536	5600	74,200	Concrete apailed at edge.
.571	6,000	79,500	. Concrete apailed at edge. I slat at juck end of ros
.604	6400	84,800	
.646	4800	90,100	
.488	7200	95,000	
			Correcte scrilled a issue all
			lateral dellection of specimen

*JACK THRUST EQUAL SHEAR LOAD ON INSERT.

JACK THRUST (LBS) = GAUGE PRESSURE (P.S.I.) TIMES /3.25

JACK: EQUIPMENT NUMBER RCH 600

PRESSURE GAUGE: M&TE NUMBER 1821 DUE DATE: 9 June 83

DIAL GAUGE: M&TE NUMBER 2094

PERFORMED BY:

DUE DATE: 20 June 83

WITNESSED BY:

- 10 Sural 3-22-93

RICHMOND 1-1/2-INCH TYPE EC-6W INSERTS SHEAR TESTS RE-ERENCE: CP-EI-13.0-8

DATE 22 Truck 83 SPECIMEN NUMBER: 5 W/O SHIM PL. Att BOLT SPEC: A - 490 W/SHIM PL. DEFLECTION | GAUGE JACK* (IN.) PRESSURE THRUST NOTES - FAILURE MODE (P.S.I.) (LBS). 0.013 400 5300 800 ,052 10.600 .091 1200 15.900 .132 1600 21 200 .180 2000 26.500 .220 2000 31,200 .265 2800 37,100 .303 3200 42,400 .336 3600 47,700 1365 4000 53,000 -391 4400 58.300 .415 4800 63,500 ,446 5200 68,900 .479 5600 Concerte spalled slightly it 74,200 .509 6000 79,500 edge of slat under fick so .538 6400 84.800 .570 6800 90,100

*JACK THRUST EQUAL SHEAR LOAD ON INSERT.

7200

JACK THRUST (LBS) = GAUGE PRESSURE (P.S.I.) TIMES 13.25

JACK: EQUIPMENT NUMBER RCH 606

PRESSURE GAUGE: M&TE NUMBER 1821

95,400

DUE DATE: 9 June 83

tat insert descritting extrac mor

ment of insert.

DIAL GAUGE: M&TE NUMBER 2094

DUE DATE: 20 June 83

pelet in local spalling of concerte

PERFORMED BY:

.616

WITNESSED BY:

J. G. Gilbett 3-22-83

RICHMOND 1-1/2-1NCH TYPE EC-6W INSERTS SHEAR TESTS

REFERENCE: CP-EI-13.0-8

OLT SPEC:	GAUGE	JACK*	SHIM PL. W/O SHIM PL.
(IN.)	PRESSURE (P.S.I.)	THRUST (LBS).	NOTES - FAILURE MODE
0.034	400	5,300	
.067	800	10,600	
. 099	1200	15,900	
. 134	1600	21,200	
.173	2000	26,500	
.225	2400	31,800	
.284	2800	37,100	
352	3200	42,400	
,207	3600	47,700	建筑的中央市场上发现了
.442	2000	53,000	
.400	4400	58,300	
.624	4800	63,600	原本的企业的企业的企业的企业的
.674	5200	68,900	Concrete sighlie at upon
.725	5600	74,200	Slat at jack end of rods
.765	6000	79,500	
.809	6400	84,800	1000000000000000000000000000000000000
.855	6800	90,100	是是學學學學學學學學學學學學學學學學學學學學學學學學學學學學學學學學學學學學學
			Concrete sincia locally
		THE PARTY	around insirt - allowing
			Cateral dellection of expen
			Last of is set
ACK THRUST	EQUAL SHE	AR LOAD ON	INSERT.
ACK THRUST	(LBS) = G	AUGE PRESSU	RE (P.S.I.) TIMES 13.25
ACK: EQUI	PMENT NUMBI	ER 1901	606
RESSURE GAL	UGE: M&TE	NUMBER /	1821 DUE DATE: 9 Lene 83
IAL GAUGE:	M&TE NUM	BER 4	2094 DUE DATE: 20 jan 83
PERFORMED BY	Y:		WITNESSED BY:

RICHMOND 1-1/2-INCH TYPE EC-6W INSERTS SHEAR TESTS

REFERENCE: CP-EI-13.0-8

	4 36 R		SHIM PL. W/O SHIM PL. HT
EFLECTION (IN.)	PRESSURE (P.S.I.)	JACK* THRUST (LBS).	NOTES - FAILURE MODE
0.081	400	5,300	
.272	800	10,600	
.400	1200	15,900	
.483	1600	21,200	
,516	2000	26,500	
.568	2400	31,800	
.465	2800	37,100	
.732	3200	42,400	
.81.	36.00	47,700	
.883	2000	53,000	
	4400	58300	Dist gauge dottened out
1			all discernable deflections were deformation of Fall
		AR LOAD ON AUGE PRESSU	INSERT. RE (P.S.I.) TIMES 13.25
ACK: EQUI	PMENT NUMB	ER KCH	606
RESSURE GA	UGE: M&TE	NUMBER /	821 DUE DATE: 9 June 83
IAL GAUGE:	M&TE NUM	BER 2	094 DUE DATE: 20 June '83
ERFORMED B	Y: Tilbin 3-2:	*/	WITNESSED BY:
7,0,	2 2	- 0-	1/200 MCD 24 3-22

RICHMOND 1-1/2-INCH TYPE EC-6W INSERTS SHEAR TESTS REFERENCE: CP-EI-13.0-8

ECIMEN NUMBER: 8 DATE 22 March 83

SPECIMEN NUMBER: 8 BOLT SPEC: SA 36 ROD W/SHIM PL. W/O SHIM PL. DEFLECTION GAUGE JACK* NOTES - FAILURE MODE (IN.) PRESSURE THRUST (P.S.I.) (LBS). 5,300 0.029 400 -190 800 10,600 .345 1200 15,900 1600 .408 21.200 2000 26,500 .457 31,800 .526 2400 418 2000 37,100 .698 3200 42,400 .745 3600 47,700 .815 4000 53,000 .890 4400 58.300 .992 63,680 4800 bet, it being deformed

*JACK THRUST EQUAL SHEAR LOAD ON INSERT.

JACK THRUST (LBS) = GAUGE PRESSURE (P.S.I.) TIMES 13.25

JACK: EQUIPMENT NUMBER RCH 606

PRESSURE GAUGE: M&TE NUMBER 1821

DIAL GAUGE: M&TE NUMBER 2094

PERFORMED BY:

J. C. Hillioth 3.22-83 DUE DATE: 9 June 83

DUE DATE: 20 June 83

WITNESSED BY:

QA REPRESENTATIVE DATE

RICHMOND 1-1/2-INCH TYPE EC-6W INSERTS SHEAR TESTS REFERENCE: CP-EI-13.0-8

SPECIMEN NUMBER: 9

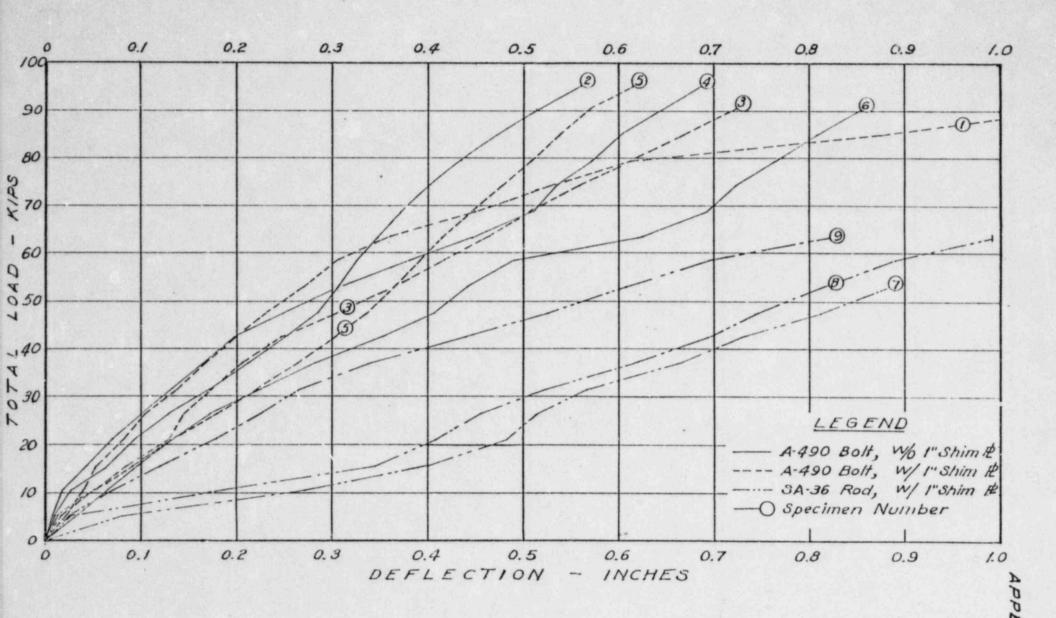
BOLT SPEC: 54-34 Red W/SHIM PL. W/O SHIM PL.

EFLECTION	GAUGE	JACK*	10752
(IN.)	PRESSURE (P.S.I.)	THRUST (LBS).	NOTES - FAILURE MODE
2027	400	5,300	PAR CONTRACTOR OF THE STATE OF
.07/	800	10,600	
.120	1200	15,900	
.179.	1600	21,200	
. 225	2000	26,500	
-266	2400	31,800	
.340	2800	37,100	
.440	3260	42,400	
1526	3600	47,700	
609	4000	53,000	
698	4400	58,300	
,821	4800	63,600	对影响的 医神经神经神经神经神经神经神经神经神经神经神经神经神经神经神经神经神经神经神经
			deformation of loct. m
			deformation of bott. m
	FORETER		sign of failure of Bolt on
			deformation.
	ROSE IN		

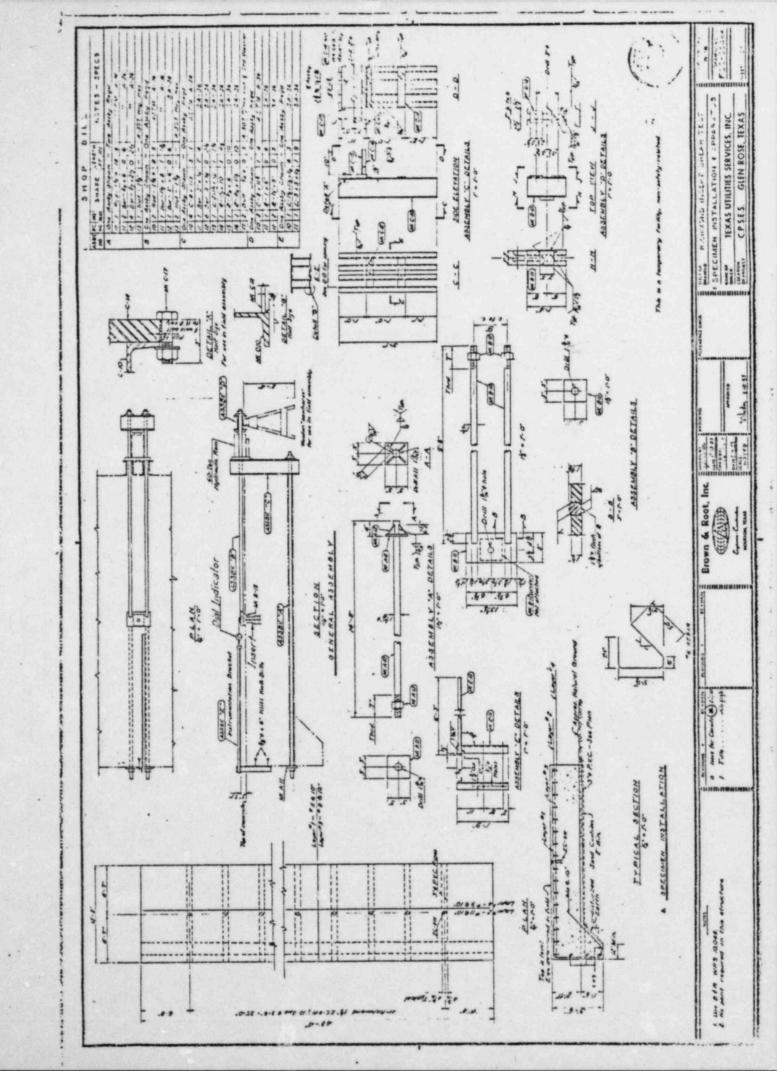
JACK THRUST EQUAL SHEAR LOAD ON INSERT JACK THRUST (LBS) = GAUGE PRESSURE (P.	
JACK: EQUIPMENT NUMBER RCH GO	,
PRESSURE GAUGE: M&TE NUMBER 1821	DUE DATE: 9 June 83
DIAL GAUGE: M&TE NUMBER 2094	DUE DATE: 25 June '83
PERFORMED BY:	WITNESSED BY:

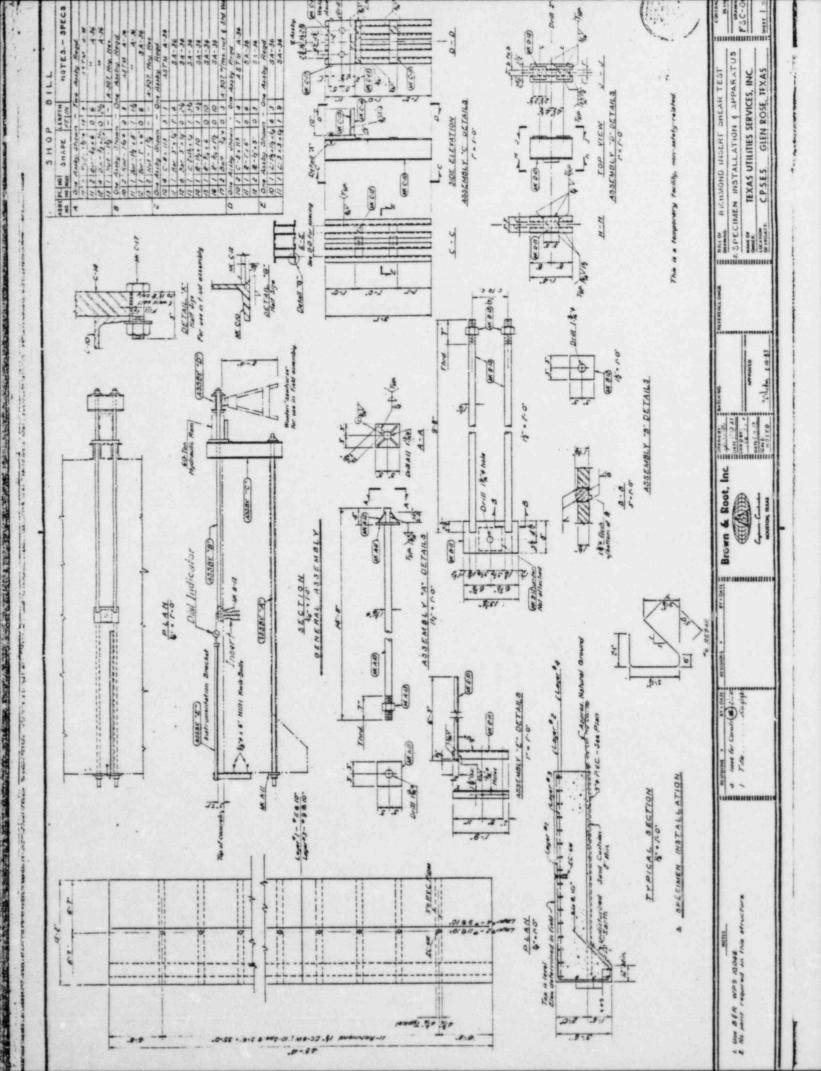
3-22-83 DATE APPENDIX 3

LOAD-DEFLECTION CURVES



LOAD - DEFLECTION CURVES





ATTACHMENT B

TEST REPORT

SHEAR AND TENSION LOADING OF RICHMOND INSERTS 1 1/2-INCH TYPE EC-6W 1-INCH TYPE EC-2W

APRIL 19, 1984

Prepared by

S.G. McBee

Civil Engineer

Approved by

R.M. Kissinger JP.E. Project Civil Engineer

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 - 2.4.2 SHEAR TEST APPARATUS
 - 2.4.3 TENSION TEST APPARATUS
 - 2.4.4 COMBINED SHEAR AND TENSION TEST APPARATUS
- 3.0 TEST PROCEDURE
- 4.0 RESULTS
 - 4.1 1 1/2-INCH RICHMOND INSERTS
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 - 4.1.2 TENSION TESTS
 - 4.1.3 COMBINED SHEAR AND TENSION TESTS
 - 4.2 1-INCH RICHMOND INSERTS
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- 5.0 CONCLUSIONS

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APPENDIX 4 - PICTURES OF ACTUAL TEST APPARATUS

TEST REPORT

SHEAR AND TENSION LOADING

OF

RICHMOND INSERTS

1 1/2-INCH TYPE EC-6W

AND

1-INCH TYPE EC-2W

1.0 REFERENCES

- A CP-EP-13.0 Test Control
- B CP-EI-13.0-13 1 1/2" and 1" Richmond Insert Shear and Tension Tests
- 2.0 GENERAL

2.1 DEFINITIONS

Ulimate Load - The load applied to the specimen which caused a physical rupture of the specimen.

Failure Load - The load applied to the specimen beyond which, deflections increased considerably without substantial increase in the applied load.

2.2 PURPOSE AND SCOPE

These tests were performed to determine the characteristics of 1 1/2-Inch Type EC-6W and 1-Inch Type EC-2W Richmond Inserts when installed in concrete representative of that used in the power block structures at CPSES. The test specimens were subjected to shear, tension, and combined shear and tension loadings. The strength, deflections, and type of deformations produced by these loadings were the qualities to be determined.

2.3 RESPONSIBILITY

The tests were performed under the direction of the CP Project Civil Engineer. Witnesses to the tests were: A TUGCO site Quality Assurance representative and other site engineering personnel.

2.4 TEST APPARATUS

2.4.1 CONCRETE SLAB & EMBEDMENTS

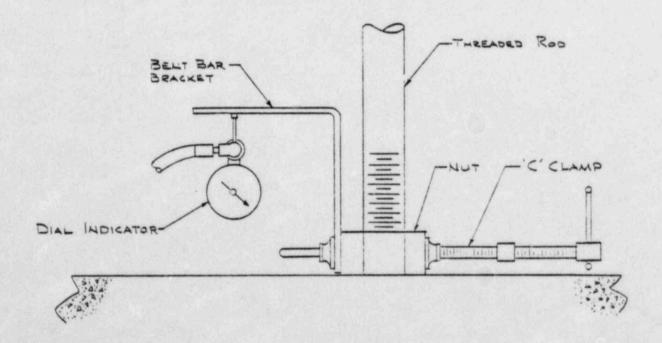
The arrangement and details of the test apparatus are shown on Drawing No. FSC-00464, Sheet 1, 2 and 3, which are included in Appendix 1 to this report. (Note that only MK C-14, C-15, C-16 and Assembly 'D' on Sheet 1 were used in this test.) The insert specimens tested were taken at random from the Constructor's stock on site and therefore, were representative of those installed in the plant structures. They were placed in a concrete slab cast specifically for these tests and which was composed of materials and reinforcement similar to those elements of the plant buildings. The concrete used was based on having a minimum design strength of 4000 pounds per square inch at 28 days. The laboratory test report on the concrete of which this slab is composed is included here in Appendix X. 2.

2.4.2 SHEAR TEST APPARATUS

An apparatus for applying shear loads to the specimens was designed and built on site. This facility employed a 60-ton capacity, manually operated hydraulic ram whose thrust against a cross head was transmitted by tension rods to a 1 1/2-inch thick shear plate bolted to the insert specimen. The base reaction of the jack was transmitted through a structural steel "bridge" to the outer face of the concrete test slab. This arrangement, as shown in Appendix 1, provided a horizontal shear load on the vertically positioned insert without producing second ry or reactive concrete stresses in the vicinity of the specimen. Ram thrust was determined by multiplying the fluid pressure (PSI), as indicated by a calibrated gauge on the pump, by a number equal to the ram piston area in square inches. Deflections were measured by a calibrated dial indicator mounted on a remotely anchored bracket and with its spring loaded probe in contact with a lug welded to the shear plate directly behind the bolt head or threaded rod.

2.4.3 TENSION TEST APPARATUS

An apparatus for applying tension loads to the specimens was also designed and built on site. This facility employed a 60-ton capacity, manually operated hydraulic ram which serves as an end loading on a built-up steel beam. The other end of the beam was bearing against a well-supported round bar which served as a fulcrum and provided the other end reaction of the beam when the jack was operated to load the specimen A threaded rod protruded through the beam at mid-span, through a nut and bearing plate on the beam with the opposite and threaded into the Richmond Insert. This arrangement caused the load on the rod to be equal to twice the force applied to the jack. Location of the base plates for the reactions of the beam provided clearance from the insert of at least 4 times the overall insert height; i.e., at least 39 1/2 inches for the 1 1/2 inch inserts and 23 inches for the 1 inch inserts. Ram thrust was determined by miltiplying the fluid pressure (PSI), as indicated by a calibrated gauge on the pump, by a number equal to the ram piston area in square inches. Deflections were measured by a calibrated dial indicator mounted on a remotely anchored bracket and with its spring loaded probe in contact with a bracket which was securely clamped to the nut on the threaded rod, as shown in the sketch below.



2.4.4 COMBINED SHEAR AND TENSION TEST

The apparatus for the combined shear and tension test utilized the same equipment as that used on the individual shear and tension tests. For the shear portion, the equipment was set up identically to the individual shear test. For the tension portion, the equipment was arranged in a slightly different fashion. The hydraulic ram was not placed under the end of the beam, but instead, on the center of the beam on top. The ram thrust was applied directly to the threaded rod, which passed through the center of the ram, by means of a plate which was placed on top of the ram. The base reaction was resisted by the tension beam, loading which was supported by two wide flange stands at sufficient distance from the insert so as not to induce secondary or reactive concrete stresses in the vicinity of the specimen. This arrangement caused the load on the rod to be equal to the ram thrust. Both rams (one applying tension and one applying shear) were operated by a single nand pump with a calibrated pressure gauge. In this fashion, the shear and tension loads applied to the test specimen would be equal at all times.

3.0 TEST PROCEDURE

In performance of all of the tests, inserts were cleaned of concrete mortar and other trash that would affect bolt thread engagement. A new bolt (A-490) or threaded rod (SA-193 Grade B7) was used for each insert. The fasteners were all tightened "snug tight". The application of all loads was applied by the ram by operation of the manual hydraulic pump. As the load increased from zero (0), indications of fluid pressure (later converted to load) and simultaneous bolt head deflection were read at regular intervals. These intervals were at 400 PSI on the pressure gauge, corresponding to 5300 pounds thrust with the exception of the direct tension tests. On the direct tension test, these intervals were at 200 PSI on the pressure gauge, which also corresponded to 5300 pounds thrust on the specimen due to the configuration used. The load as indicated by these gauge pressures was maintained as constant as possible for a period of two (2) minutes. At the end of this time period, the deflection was again observed and noted. Load application on each specimen was carried out until ultimate failure of the specimen occured (except specimen no. 1, which was tested in shear). At this point, observations were made of the condition of the specimens and the failure mode.

- 4.0 RESULTS
- 4.1 1 1/2-INCH RICHMOND INSERTS

4.1.1 SHEAR TESTS

As can be seen on the test data sheets, the ultimate load applied to the specimens ranged from 90,100 lbs, to 106,000 lbs.. The failure loads ranged from 84,800 lbs. to 106,000 lbs.. All bolts sheared abruptly (except specimen #1; test was halted prior to ultimate failure), with minor spalling of the concrete on the compression side of the Richmond Insert. All five (5) specimens were utilizing A-290 bolts.

SPECIMEN NO.	ULTIMATE LOAD (16s)	FAILURE LOAD (1bs)
1	90,100	84,800
2	95,400	90,100
3	95,400	90,100
4	106,000	100,700
5	106,000	106,000
Average	98,580	94,340

Using the allowable insert laods given in specification 2323-SS-30 for 1 1/2-inch Richmond Inserts, the factor of safety is determined.

Allowable Shear = 27.0k

Factor of Safety (F.S.) = Average Failure Ld.

Design Allowable Ld.

SPECIMEN NO.'s	AVERAGE FAILURE LOAD (k)	FACTOR OF SAFETY
1 thru 5	94.34	$\frac{94.34}{27.0} = 3.49$

4.1.2 TENSION TESTS

The ultimate load applied to the tension test specimens ranged from 87,650 lbs. to 114,150 lbs.. The failure loads ranged from 87,650 lbs. to 108,850 lbs.. The failure mode for specimens 11 and 12 was by striping the threads between the threaded rod and the Richmond Insert. Specimen 13 failed in the Richmond Insert by a failure of the welds between the axial strut rods to the upper threaded coil. Specimens 14 and 15 failed by concrete shear cone failures. All specimens were utilizing SA-193 Grade 87 threaded material.

SPECIMEN NO.	ULTIMATE LOAD	FAILURE LOAD
11	106,200	103,550
12	114,150	108,850
13	114,150	108,850
14	87,650	87,650
15	100,900	100,900
Average	104,610	101,960

Allowable Tension = 31.3k

Factor of Safety (F.S.) = Average Failure Ld.
Design Allowable Ld.

SPECIMEN NO.'s	AVERAGE FAILURE LOAD (k)	FACTOR OF SAFETY
11 thru 15	101.96	101.96/31.3 = 3.26

4.1.3 COMBINED SHEAR AND TENSION TESTS

The shear and tension loads applied to the specimens under this loading condition are equal and the ultimate loads ranged from 60,950 lbs. to 68,900 lbs.. The failure loads ranged from 58,300 lbs. to 67,575 lbs.. Specimens 6 through 9 failed by an abrupt shearing of the threaded rod. There was some deformation of the rod in bending at the shear zone (ranging for 20° to 45° bend). Upper insert washer moved from 1/2 inch to 3/4 inch with some concrete spalling on the compression side of the insert. Specimen 10 failed by striping the threads between the threaded rod and the insert. This failure lifted the upper insert washer from the struts, but the insert remained in place.

SPECIMEN NO.	ULTIMATE LOAD (1bs)	FAILURE LOAD (1bs)
6	68,900	67,575
7	67,575	67,575
8	60,950	58,300
9	61,613	61,613
10	64,925	62,275
Aver	age 64,793	63,468

Allowable Tansion = 31.3k

Allowable Shear = 27.0k

Factor of Safety (F.S.)

$$(\frac{\text{Average Failure Tension}}{\text{Design Allowable Tension x F.S.}})^{4/3} + (\frac{\text{Average Failure Shear}}{\text{Design Allowable Shear x F.S.}})^{4/3} = 1.0$$

SPECIMEN NO's.	TENSION AND SHEAR AVFRAGE FAILURE LOAD (k)	FACTOR OF SAFETY
6 thru 10	63.47	$\left(\frac{63.47}{31.3 \times F.S.}\right)^{4/3} + \left(\frac{63.47}{27.0 \times F.S.}\right)^{4/3}$
		E C = 2.62

4.2 1-INCH RICHMOND INSERTS

4.2.1 SHEAR TESTS

From the test data sheets, the ultimate load applied to the specimens ranged from 39,750 lbs. to 50,350 lbs.. The failure loads ranged from 37,100 lbs. to 42,400 lbs.. Specimens 16 thru 19 failed by shear failure of the A-490 bolt. The top portion of the inserts deflected from 1/8 inch to 7/8 inch with some spalling on the compression side of the insert. Specimen 16 showed some rotation of the top of the insert. Specimen 17 and 18 showed no apparent sign of rotation. Specimen 19 failed by breaking the weld between the upper coil and the struts. The bolt then failed in bending after rotating with the upper portion of the coil. Specimen 20 failed by crushing the concrete on the compression side of the insert. The insert then rotated intact and the bolt ultimately failed in bending.

SPECIMEN NO.	UL	TIMATE LOAD (1bs)	FAILURE LOAD (1bs)
16		46,375	42,400
17		43,060	37,100
18		50,350	42,400
19		46,375	42,400
20		39,750	37,100
	Average	45,182	40,280

Allowable Shear = 11.5k

Factor of Safety (F.S.) = Average Failure Ld.

Design Allowable Ld.

SPECIMEN NO'S.	Average Failure Load (k)	Factor of Safety
16 thru 20	40.28	40.28/11.5 = 3.50

4.2.2 TENSION TESTS

The ultimate load applied to the specimens ranged from 41,270 lbs. to 43,920 lbs.. The failure foads ranged from 39,950 lbs. to 43,920 lbs.. Specimens 26, 28 and 29 failed by concrete shear cone failure. Specimens 27 and 30 failed by Richmond Insert failure. The inserts failed by a failure of the welds between the struts and the lower coil. There was some surface spalling associated with these failures.

SPECIMEN NO.	UL	TIMATE LOAD (165)	FAILURE LOAD (1bs)
26		42,600	42,600
27		43,920	43,920
28		42,600	39,950
29		42,600	39,950
30		41,270	39,950
	Average	42,598	41,276

Allowable Tension = 11.5k

Factor of Safety (F.S.) = Average Failure Ld.
Design Allowable Ld.

SPECIMEN NO'S.	AVERAGE FAILURE LOAD (k)	FACTOR OF SAFETY
26 thru 30	41.276	41.276/11.5 = 3.59

4.2.3 COMBINED SHEAR AND TENSION TESTS

The shear and tension loads applied to the specimens under this loading condition are equal and the ultimate loads ranged from 27,825 lbs. to 30,475 lbs.. The failure loads ranged from 27,825 to 29,150 lbs.. Specimens 21 thru 25 failed abruptly due to shear failure of the threaded rod. All inserts remained intact with only surface spalling of the concrete.

SPECIMEN NO.	ULTIMATE LOAD (16s)	FAILURE LOAD (1bs)
21	27,825	27,825
22	29,150	29,150
23	30,475	29,150
24	29,150	27,825
25	28,487	27,825
Averag	e 29,017	28,355

Allowable Tension = 11.5k

Allowable Shear = 11.5k

Factor of Safety (F.S.)

SPECIMEN NO'S	TENSION AND SHEAR AVERAGE FAILURE LOAD (k)	FACTOR OF SAFETY
21 thru 25	28,355 (_T	$\frac{28.36}{1.5 \times F.S.}$)4/3+ $(\frac{28.36}{11.5 \times F.S.})$ 4/3
		=1.0

F.S. = 4.15

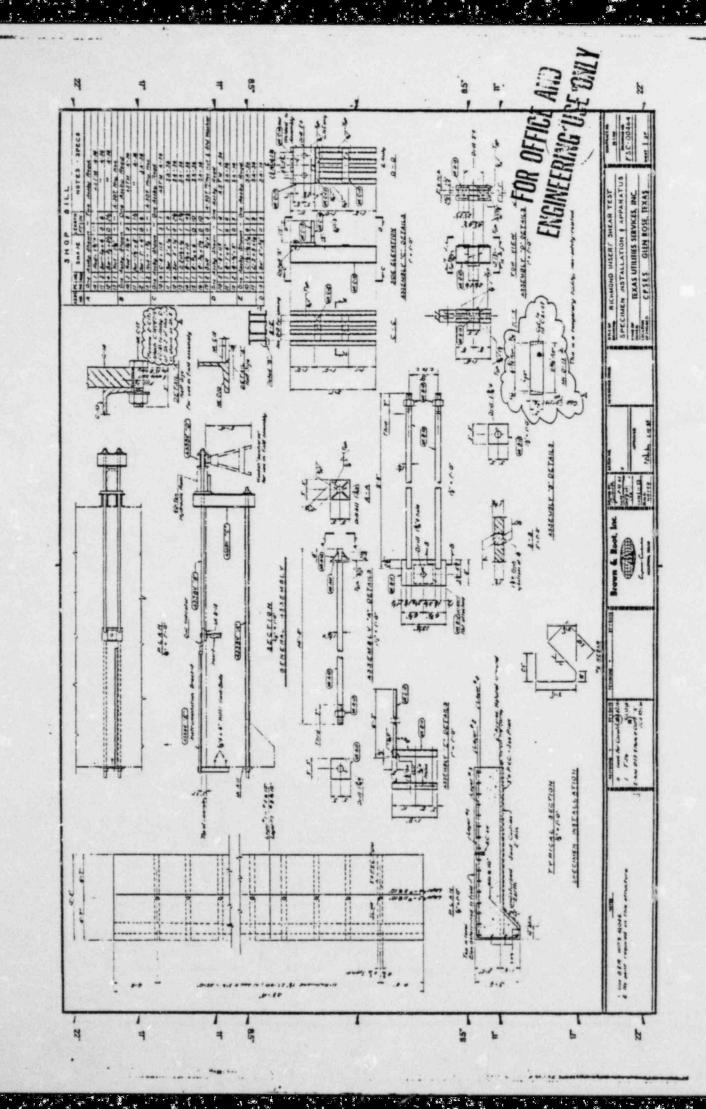
5.0 CONCLUSIONS

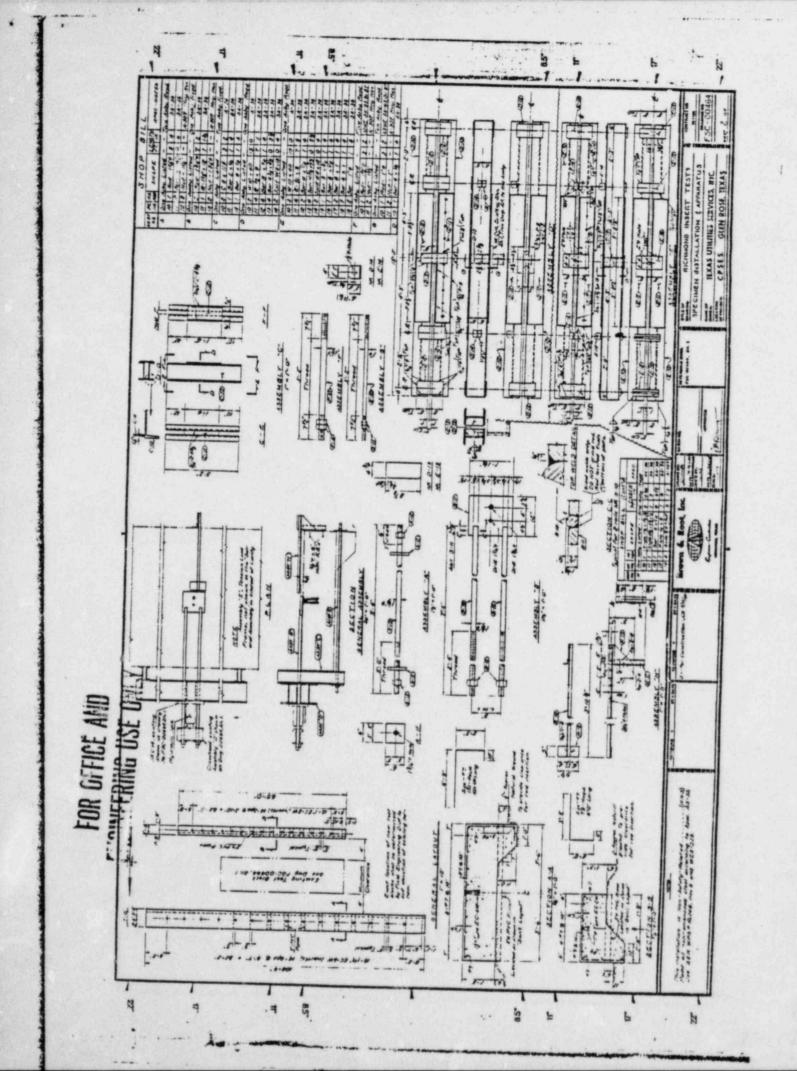
These test results show that the performance capabilities of the 1 1/2-inch type EC-6W and the 1-inch type EC-2W Richmond Inserts in shear, tension and combined shear and tension exceed the design allowable by a ratio of more than 3 to 1. These conclusions are valid for the design allowables shown in Specification 2323-SS-30, based on a spacing of the Richmond Inserts such that a full shear cone can develop.

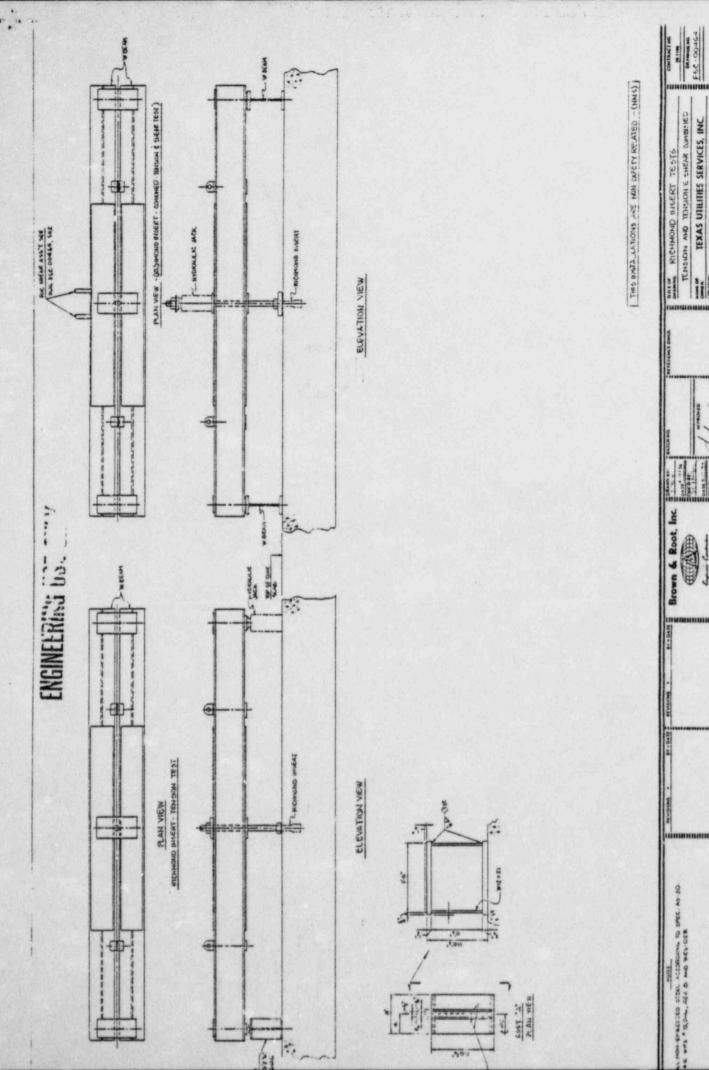
Based on this test, the design allowables for shear, tension and combined shear and tension are acceptable for use without further investigation or additional calculations. Richmond's recommendation of a minimum safety factor of 3 has been complied with.

APPENDIX 1

DRAWING NO. FSC-00464 SHT. 1, 2 & 3







CPSES GIEN ROSE, TEXAS

APPENDIX 2

CONCRETE COMPRESSIVE TEST REPORT

							COMANC	HE PEAK	SES		2-2	19-89	
POI	RT ON CO	MPRE	SSIVE TE	STS OF C	ONCRETE					CYL. SET	0/7-	473	4-00/
OCE	DURE 4	1-0	7P-16	(-4/						- (Test .	Block)
	COMPLET OATA AS		AGGR	9408	H201		85 13560	1420 C A	Les	TAL WATER	BATCH T	MY	WAING
Ä	FROM BAT		SOG 1	YO L85	H20 A006	GAL	0.38	HATIO AIR 11.42	Y2.	- U. I	YOOO	DESIGN S	~
NAL S	BRAND OF CEMENT TYPE OF CEMENT BRAND OF AIR ENTRAINING ADMIXTURE BRAND OF WA									NA REDUCING	ADMIXTUR	-	SIZE CA
MATE	SOURCE C		TOP	A	CA CA 2.65		ACE FA	TOP	SP GA F.		FINENESS 2.		S FA
	TYPE OF MI		BATCH LOAD		TICKET NO.	and the same of the same of	E TAKEN			/			
9	PLAN	TI	8	c. 4	2989		CENTRAL	HIXER [FORMS		ONT OF D	NSCHARGE	
MPLIN	METHOD BUGG		LACING .	PUMP CHUT	BUCKET DAT	29-84	CAN	CLEAR	AIR TE	F CONC.	O .F	33	4.
3				UNIT WT.			1	SPECIMEN	74	CONTO	NEN CAST	87	AND .
	CENTRAL	PLANT			4.36		132	BIRCHI			D0 -Z		5.0 %
10	LINDER ID.	KEV	MEASURED		DATE	CAPPED	TIME	DATE	-	COMPRESSM	T C.A.		
-			6.980	IN.	CAPPED	BY .	TESTED	TESTED	LG.	STRENGTH	- av	87	DREAK
CURED	2473 H	7	6.000	5.990	3684			3-7-84			100	02	Real
0	2473 8	1	5.9%	6.001	3-1-84			3-7-84			uc		
3	2473C	28		1000	3.23 84	Sm	2702-	3-28-84	153500	5430	100	w	Pon
	2473 D	28	600	10008	3.238	2W	0648	3-28-84	155500	5490	w	er	H ag
	2473E	28	6.001	10.002	3:33	Sm	10659	3-28-84	139500	4970	4	evi	Reg
CUNEC	2473F	28	5.991	4000	3-23-81	500	0656	3-28-84	141000	4930	CIT	UT	Real
9	NA												-
-	NA												
	3-1-84	IME .	STRIPPED	REMARKS									
			RATORY CL	URED CYLIN			ROL TEST			D CYLINGER			
		STRE	NGTH (PS.I.)	5430			5	TRENGTH I	P.S.I)		4930	
		1 (0)	-(b) - (c)	*/d) *	0,91	(d) *		,	(a)+(b)	- 2 -		4980	
		NOT	NOT	EXCEED TO		STRENGTH		N 0.85; OR					
			MJ-TE		1/21	,	DAY PREPA	AED 37	-vic-	CHECK	ED AY	TA	5
			HINE NO					4ED BY		OHECK		IA	
IGN.	ENGINEERS	COM	MENTS UF A	PRICARIE!							7	77	

APPENDIX 2

TEST DATA SHEETS

COMANCHE PEAY SES

SHEAR TESTS

RICHMOND 1/2 - INCH, TYPE INSERT

Reference: CP-EI-13.0-全/3gcは

Specimen Number: / Bolt Spec: A-490 Date: 3 Apr. 44

(First insert @ west end of conc. s/ab)

DEFLECTION (IN.) GAUGE JACK *

DEFLECT	ION (IN.)	GAUGE	JACK *	
INITIA	AFTER	PRESSURE	THRUST	NOTES-FAILURE MODE
	2-MIN.	(P.S.I.)	(Lbs.)	
0.003	0.003	\$ 400	5300	
.032	.035	800	10600	
,060	.060	1200	15 900	
.076	.079	1600	2/200	
.095	-098	2000	26500	
.111	. 116	2400	3/800	
.128	.132	2800	37 100	
.144	./50	3200	42 400	
.160	167	2600	47700	
-178	.185	2000	53000	
.196	.206	4400	58300	
.220	.233	4 800	63 600	
,250	,24	5200	68 900	
.277	.297	5600	74 200	
.304	.348	6000	79 500	Bolt deformed.
380	.429	6400	84 800	Crushing of concrete was
510	1.125	6800	90 100	orincipal failure. No increase
	Harasa II			in load with increased deflec-
			LEASE TO S	tion. Did not load to destruction
	I managed to	Burned of	A bolthead	for removal. Insert stayed fast
		in concre		

Jack Thrust equal Shear Load on Insert.

Jack Thrust (Lbs.) = Gauge Pressure (PSI) x 13.25

Jack:.....Equipment Number RCH 606

Pressure Gauge: M & IE Number 2355

Due Date: 16 con 199

Dial Gauge:....M & TE Number 2949

Due Date: 29 Jane 84

Performed By:

O. Gilbrett 3 april 84

Alex Prity & 4-3-84
Representative Date

Witnessed By:

COMANCHE PEAK SES SHEAR TESTS

EC-EW RICHMOND / - INCH, TYPE INSERT

Reference: CP-EI-13.0-7/1 gch

Specimen	Number:	2	Bolt	Spec:	A-490	Date: 4	april	89
		STATE OF THE PARTY				-		_

DEFLECTION	ON (IN.)	GAUGE	JACK *	Warra 5111 Was word
INITIAL	AFTER	PRESSURE	THRUST	NOTES-FAILURE MODE
	2-MIN.	(P.S.I.)	(Lbs.)	
1.002	0.002	400	5,300	
.021	,022	800	10,600	
.072	. 036	1200	15,900	
.049	.051	1600	21,200	
.063	.066	2000	26,500	
.080	.083	2400	31,800	
.096	.102	2800	37 100	
.115	.121	3200	42 400	
.133	.142	3600	47700	
.157	. 166	4000	53 000	
.180	.192	4400	58 300	
, 208	.217	4800	63.600	Bank in the tradelinearity as per set with the east
.237	.247	5200	68,900	
.263	.276	5600	74,200	
.293	.314	6000	79,500	
.338	.370	6400	89,800	
, 480	.555	6800	90 100	
. 770	1.110	7200	95,400	Boit sheared abruptly. Concrete
1				spalled on compression side of inser
	Aprico	15" duep 1	inning out	to zero @ T'sway. Swall app. To
		de near ms		

Dial Gauge:..... M & TE Number 2949 Due Date: 29 Jun 84

Performed By:

C. Hillath

Witnessed By:

COMANCHE PEAK SES

SHEAR TESTS

EC-EW RICHMOND / - INCH, TYPE INSERT

Reference: CP-EI-13.0-X /3 yes

Specimen Number: 3 Bolt Spec: 4-490 Date: 4 April 84

(3d from West End)

DEFLECTION	ON (IN.)	GAUGE	JACK *	
INITIAL	AFTER	PRESSURE	THRUST	NOTES-FAILURE MODE
	2-MIN.	(P.S.I.)	(Lbs.)	
0.000	.0.	400	5300	
.002	.002	800	10 600	
,003	.003	1200	15 900	
.006	.007	1600	21200	
.0/2	.018	2000	26500	
.032	.036	2400	31800	
.049	.052	2800	37100	
.067	.069	3200	42 400	
.078	.083	3600	47700	
,096	.107	4000	53 000	
.126	.131	4400	58300	
,144	.154	4800	63600	
.174	.182	5200	68 900	
.206	.218	5 600	74 200	
.242	.259	6000	79 500	
.283	.315	6400	84800	
.365	,399	6800	90/00	
,540	(1.2)	7200	95 400 b	Boit sheared abruptly concre
		palled I"de		ort, tapering to zero depth
				on side of insert). Insert
		and the second s		4). Insert seeminally intact where still

Jack Thrust equal Shear Load on Insert. Jack Thrust (Lbs.) = Gauge Pressure (PSI) x /3,25

Jack:.....Equipment Number RCH 606
Pressure Gauge: M & TE Number 2355
Dial Gauge:.....M & TE Number 2949

Due Date: 16 Apr 84

Witnessed By:

Due Date: 29 yun 84

deflected abt 5/8

Performed By:

9. Gilbert 4 april 84

COMANCHE PEAK SES SHEAR TESTS

RICHMOND 16-INCH, TYPE INSERT

Reference: CP-EI-13.0-X/3 yes

Specime Number: 4 Bolt Spec: A - 490 Date: 4 April 84

(4th from West End)

DEFLECTI	ON (IN.)	GAUGE	JACK *	
INITIAL	AFTER	PRESSURE	THRUST	NOTES-FAILURE MODE
	2-MIN.	(P.S.I.)	(Lbs.)	
.0005	.0005	400	5,300	
.003	,003	800	10,600	
,012	.013	1200	15,900	
.024	.026	1600	21,200	
.035	,038	2000	26,500	
.047	.048	2400	31,800	
.058	. 059	2800	37,100	
.067	.070	3200	42,400	
.078	.081	3600	47, 700	
.089	.094	4000	53,000	
.103		9400	_	accidental opening of review - Torch ristorch
107	.109	4600	58,300	
.116	.120	4800	63 600	THE RESERVE OF THE PARTY OF THE PARTY.
./28	.133	5200	68,900	
.142	-146	5600	74,200	
.156	.164	6000	79,500	
./73	.181	6400	84.800	
,292	.303	6800	90,100	
.3/5	.333	7200	95,400	
. 360	.389	7600	100,700	
.550		8000	106,000	Boit sheared abrustly. Concrete spall

*	Jack Thrust equal Shear Load on Insert. I" deep & Insert to O" of 4" out. Spall &" wide	
	took Though (I be) - Course Descript (DCT) v 12 35	
	Jack:Equipment Number RCH 606	
	Pressure Gauge: M & TE Number 2355 Due Date: 16 Apr 84	
	Dial Gauge: M & TE Number 2949 Due Date: 27 Jun 84 Insert defla	

Witnessed By:

Performed By:

Q.C. DilRett 4 april 84

Achen active Date

COMANCHE PEAK SES SHEAR TESTS

RICHMOND /= INCH, TYPE INSERT

Reference: CP-EI-13.0-X/2ge#

Specimen Number: 5 Bolt Spec: 4-490 Date: 4 April 84

(5th from West End)

DEFLECT	ION (IN.)	GAUGE	JACK *	
INITIAL	AFTER	PRESSURE	THRUST	NOTES-FAILURE MODE
	2-MIN.	(P.S.I.)	(Lbs.)	
0.002	0.002	400	5 300	
,004	.005	800	10 600	
.013	.015	1200	15 900	
,035	.037	1600	21 200	
,057	,063	2000	26500	
.090	,094	2400	31 800	
.117	.124	2800	37 100	
.150	.157	3200	42 400	
.176	.183	3600	47 700	
.200	.209	4000	53 000	
, 223	.236	4400	58.300	
.248	.261	4800	63,600	
.276	.295	5200	68,900	
,307	,322	5600	74,200	
.338	356	6000	79,500	
.370	.389	6400	84,800	
.408	.428	6800	90,100	
.447	.479	7200	95,400	
.506	.556	7600	100 700	
.58,-		8000	10 Bolt shes	red abruptly. Concrete spelled

* Jack Thrust equal Shear Load on Insert.

Jack Thrust (Lbs.) = Gauge Pressure (PSI) x 13.25

Jack:.....Equipment Number RCH 606

Pressure Gauge: M & TE Number 2355

Due Date: 16 Aor 89

Dial Gauge:....M & TE Number 2949

Due Date: 29 Jun 84

Witnessed By:

Performed By:

LC. Hilketh 4 apr 84

Adrew Pritink 4-4-84

COMANCHE PEAK SES Richmond // Inch. Type Insert
Reference: CF-E1-13.0-\$ /756~

Specimen Humber: 6 (6th from West)

Inserted Load Rod: A-193

Date: 10 April 84

Common	SHEA	8				TENSI	ON			
Gauge Press. (PSI)	Jack Thrust (Lb.)		ction ch) After 2-Min.	Gauge Press. (PSI)	Jack Thrust (Lh.)	Net Jack Thrust (Lb.)	Insert Load (Lb.)	(In	ction ch) After 2-Min.	Notes - failure Mode
400	5 300	0.007	0.007	5	5500	7	1	0.0015	0.00/5	
800	10 600	.023	,024	7	10 600	7	1	.005	.005-	
1200	15 900	.091	1042		15 900	3	5	.0095	.0/05	
1600	21 200	The second second	STATE OF THE PARTY		21 200	5	5	-018	.019	
2 000	26 500	.088	,095	1	26 500	5	5	.031	,034	
2400	31,800	.146		,	31,800	1		,046	.048	
2800	37,100	.192	.199	NA	37,100	N/A	1/4	1.054	.056	
3200	42400	, 236	.246		42,400		1	.062	,0635	
3600	47.700	,290	.304	-	47,700	5	5	0715	.074	
4000	53,000	, 339			53,000		5	,083	.087	
4250	56,313	,420			56,313		5			
4300	56,975	,460			56,975		-	Editor		
*400	58300	.475	.559		58,500	7	5	.115	.139	
4500	5964		,58	}	59,625)	1			
4600	60.950	.630			60,950	1	1			

1-* Jack Thrust = Shear Load on Insert.

1-* Jack Thrust (Lb.) = Gauge Pressure (PSI) x 13.25 for Shear Load.
2-* Jack Thrust (Lb.) = Gauge Pressure (PSI) x 13.25 for Tension Load.
Total Wt. of Tension Load Beam = NA Lb.

** Net Jack Thrust = Total Thrust Minus 1/2 Wt of Beam.

** Insert Load = *Net Jack Thrust - 2- 23 N

Shear Apparatus: Jack---Equipment No: ACH 606
Pressure Gauge-M&TE No: 2355 Due Date: 16 Que 84
Dial Gauge-M&TE No: 2949 Due Date: 29 Van 84
Tension Apparatus: Jack-Equipment No: ACH 6037
Pressure Gauge-M&TE No: Same Due Date:
Dial Gauge-M&TE No: 2094 Due Date: 18 Jun 84

Witnessed By: John Welink 4-13-84

OA Representative Date

COMANCHE PEAK SES Richmond / - Inch, Type Insert
Reference: CP-E1-13.0-9 /32c+

Specimen Number: 6 (R2 of 2) Inserted Load Rod: A-193 Date: 10 april 32 Common SHEAR TENSION 1-* ** *** Gauge Jack Deflection Gauge Jack Net Insert Deflection Press. Thrust (Inch) Press. Thrust (Inch) Jack Load Notes - Failure Mode After Thrust After (PSI) (Lb.) Init. 2-Hin. (PSI) (Lb.) (Lb.) (Lb.) lait. 2-Min. 4800 63600 ,750 .808 63 600 .167 -164 66 250 ,840 5000 66 250 67 575 .910 51005 67 575 68 900 5-200 48 900 1.0% o 0186 47 575 5100 67575 Abrupt shearing of Rod. Rod deformed in bending a shear dane, carrying insert washer some 12" horizonfally. 1-* Jack Thrust - Shear Load on Insert.

Jack Thrust (lb.) = Gauge Pressure (PS1) x 13.25 for Shear Load.

2-* Jack Thrust (lb.) = Gauge Pressure (PS1) x 13.25 for Tension Load.

Total Wt. of Tension Load Beam = N/A lb.

** Net Jack Thrust = Total Thrust Minus 1/2 Wt. of Beam.

*** Insert Load = Net Jack Thrust = 2. 2044

Performed By: J. C. Gilbell, 4-10-84

Shear Apparatus: Jack --- Equipment No: RCH 606

Pressure Gauge-M&TE No: 2355 Due Date: /c apri pur Dial Gauge-M&TE No: 2949 Due Date: 29 due 34 Tension Apparatus: Jack-Equipment No: ACN 6037 Pressure Gauge-M&TE No: Same Due Date:

Dial Gauge-M&TE No: 2094 Due Date: 18 der 84

Witnessed By: Alen Pringle 4-13-84

COMANCHE PEAK SES COMBINED SHEAR & TENSION TESTS Richmond //2-Inch, Type Insert
Reference: CP-EI-13.0-9-/1904

Specimen Number: 7 (7th from west end) Inserted Load Rod: A-193

Date: 11 April 84 Commai SHEAR TENSION 1-* ** *** Gauge Jack Deflection Gauge Jack Net Insert Deflection Press. Thrust (Inch) Press. Thrust Jack (lach) Load Notes - Failure Mode After Thrust After (PSI) (Lb.) lait. 2-Min. (PS1) (Lb.) (Lb.) (Lb.) Init. 2-Min. 400 5 300 0.000 0.000 5 300 0.000 0,000 800 10 600 .000 10 600 .009 003 .003 1200 15 900 .023 15 900 .026 012 .012 21200 1600 .041 21 200 .043 .022 .023 . 070 . 725 . 623 . 233 . 345 . 410 2000 26 500 .0765 26 500 039 31 800 37 100 42 900 41 700 53 040 58 340 .043 3600 31 800 ,063 .065 .086 .093 .129 37 190 42 400 47 700 53 000 58 300 NA N/4 14/4 165 4600 60950 .58 60 950 to failing of bracket with bending rod. .195 4700 62275 .770 62275 .210 4800 63600 802 846 r & Rod failed in shear. Top of insert 63600 .206 .198 4950 65588 .870 65 588 deflected horizontally some 5/8" Rod dis-,900 5000 66250 torted to some 30° from vert @ shear gone 66 250 505-66 913 1.15 66 913 Concrete spalled slightly around insect 1.200 5100 67 575 67 575 Breakl insert intact

1-* Jack Thrust = Shear Load on Insert.

1-* Jack Thrust (Lb.) = Gauge Pressure (PSI) x 13.25 for Shear Load.

2-* Jack Thrust (Lb.) - Gauge Pressure (PSI) x 13.25 for Tension Load.

Total kt. of Tension Load Beam = N/A Lb.

Total Wt. of Tension Load Beam = NA Lb.

** Het Jack Thrust = Total Thrust Minus 1/2 Wt. of Beam.

*** Insert Load = Not Jack Thrust-x-2 QC #

Shear Apparatus: Jack --- Equipment No: RCM COG

Pressure Gauge-MATE No: 2355 Due Date: 16 ga & Dial Gauge-M&TE No: Due Date: 29 dun 84 2949

Tension Apparatus: Jack-Equipment No: RCH 6037
Pressure Gauge-M&TE No: Jack Due Date:

Dial Gauge-M&TE No: 2094 Due Date: 18 Jun 89

Witnessed By: Ale Ordinal 9-11-94

Performed By: Q. G. Helhett, 4-11-84
Name Date

COMMANCHE PEAK SES
COMBINED SHEAR & TENSION TESTS
Richmond / -Inch. Type Insert
Reference: CP-E1-13.0-\$1394

Specimen Number: 8 (8th from west ens) Inserted Load Rod: A - 19.3 Date: 11 april 84 Common SHEAR TENSION. 1-0 ** *** Gauge Jack Deflection Jack Gauge Net Insert Deflection Press. Thrust (Inch) Press. Thrust Jack Load (Inch) Notes - Failure Mode After Thrust After (PSI) (Lb.) Init. 2-Hin. (PSI) (Lb.) (Lb.) 2-Min. (Lb.) Init. 400 5340 0.000 0,000 5300 0,000 0,000 10 600 800 .021 .021 10 600 001 .00/ /200 /600 2000 2800 3200 3600 4400 15900 27200 26500 27000 37700 15 900 .0095 0045 21 200 .008 59 588 37 100 42 400 47 700 53 000 18 340 N/A M/A 22300 9400 5 8300 58 300 51625 4500 Dial indicators removed to prevent demege 59 625 60150 4600 e time of failure. 60 750 Break! Shear failure of rod Rod bant 4550 60288 60208 about \$ 30° & shaar jone. Washer of insert moved horizontally 1/2". 2 struts broke & wold from washer. Concrete spalled I"deep to soro & s'out on comp side of insert 1-* Jack Thrust = Shear Load on Insert. Shear Apparatus: Jack --- Equipment No: RCH 606 1-* Jack Thrust (Lb.) - Gauge Pressure (PSI) x 13,25 for Shear Load.
2-* Jack Thrust (Lb.) - Gauge Pressure (PSI) x 13,25 for Tension Load.
Total Wt. of Tension Load Beam - MA Lb. Pressure Gauge-MSTE No: 2355 Due Date: 16 gar 80 Dial Gauge-MSTE No: 2949 Due Date: 28 dun 84 Tension Apparatus: Jack-Equipment No: 8CH 6037 Pressure Gauge-MSTE No: 6222 Due Date: Dial Gauge-MSTE No: 2094 Due Date: 18 Jun 84 Total Wt. of Tension Load Beam - 7/A Wt. of But Sack limest - Total Thrust Hims 1/2 Wt. of But Het Jack limest - Total Thrust + 2

Performed By: J.C. Hillstl. 4-11-84

Witnessed By: Afen Orchyl 4-11-84

COMANCHE PEAK SES COMBINED SHEAR & TENSION TESTS Richmond 12-Inch, Type Insert Reference: CP-E1-13.0-5/3 pat

Specimen Humber: 9 (93 from west and) Inserted Load Rod: A-193 Date: 11 april 84 SHEAR Common TENSION. 1.* ** *** Jack Gauge . Deflection Jack Gauge Net Insert Deflection Press. Thrust (lach) Press. Thrust Jack Load (Inch) Notes - Failure Mode After Thrust After (PS1) {Lb.} Init. 2-Min. (PSI) (Lb.) (Lb.) 2-Min. (Lb.) lait. 5 300 400 5 300 0,000 0,000 0,000 0,000 .000 .000 4600 60950 14/A .720 1/19 Dial gages removed to prevent damage 4650 61613 .77 61,613 46 375 3500 46,375 shear failure of rod. Rod shear Broak 1 zone rotated about 45° struts on comp, + tonsion sides of insert broke loose from nasher @ wolds Washer moved 14" horizontally. Insert below washer severed to be intact but threates coil slightly distorted. Conc spalled 1" x 6" on coms 1-* Jack Thrust = Shear Load on Insert. Shear Apparatus: Jack --- Equipment No: RCH 606 Pressure Gauge-M&TE No: 2355 Due Date: 16 Apr. 89
Dial Gauge-M&TE No: 2949 Due Date: 29 Von 89
Tension Apparatus: Jack-Equipment No: RCH 6037
Pressure Gauge-M&TE No: 6037
Pressure Gauge-M&TE No: 2094 Due Date: 18 Jun 89 1-* Jack Thrust (Lb.) = Gauge Pressure (PSI) x 13.25 for Shear Load.

2-* Jack Thrust (Lb.) = Gauge Pressure (PSI) x 13.25 for Tension Load.

Total Wt. of Tension Load Beam = 1/4 lb.

** Ret Jack Ibrust - Total Thrust Minus 1/2 Wt. of Bea

Performed By: Laffilbill 4-11-84
Name Date

Witnessed By: Ale Out of 4-11-34

COMANCHE PEAK SES COMBINED SHEAR & TENSION TESTS Richmond 12 - Inch. Type Insert Reference: CP-E1-13.0-9 13904

Specimen Number: 10 (10th from west end) Inserted Load Rod: A-193 Date: // april 84 SHEAR Common **TENSION** 1.* **Gauge** Jack Deflection Gauge Jack Net Insert Deflection Press. Thrust (Inch) Jack Press. Thrust Load (Inch) Notes - Failure Mode After Thrust After (PSI) (Lb.) Inic. 2-Hin. (PSI) (Lb.) (Lb.) (Lb.) Init. 2-Min. 0.000 400 0.000 0.000 5 300 5 300 800 10 600 .000 2000 2400 2800 3200 3600 4000 4100 Reat Tonsia dial due to its fouling beam 58 300 58 300 4600 60950 270 NA 60 950 N/A N/A 47 = 0 62 275 310 62 375 316 4800 63600 .145 63 COO 4800 63 600 .550 63 600 4900 63 600 .600 .625 .245 .230 65600 Threads stripped. Littled insert washer loose from struti. Insert remaned in place. 4900 64925 64 925 1-* Jack Thrust = Shear Load on Insert. Shear Apparatus: Jack --- Equipment No: RCH 606

1-* Jack Thrust (Lb.) - Gauge Pressure (PS1) x 13.25 for Shear Load.
2-* Jack Thrust (Lb.) - Gauge Pressure (PS1) x 13.25 for Tension Load.
Total Wt. of Tension Load Beam - N/A Lb.

** Het Jack Thrust - Total Thrust Minus 1/2 Wt. of Beam.
*** Insert Load - Het Jack Thrust - 2.

Performed By: J. C. Hilbett 4-11-84

Pressure Gauge-M&TE No: 2355 Due Date: 16 Ar 84
Dial Gauge-M&TE No: 2949 Due Date: 29 Jun 84

Tension Apparatus: Jack-Equipment No: RCH GO3T

Pressure Gauge-M&TE No: 2094 Due Date:
Dial Gauge-M&TE No: 2094 Due Date: 18 Jun 84

Witnessed By: Ale Gittine 4-11-84

COMANCHE PEAR SES TENSION TESTS

RICHMOND 1/2-INCH, TYPE INSERT

Reference: CP-EI-13.0-3/3 Load Rod Spec: A - 193 Date: 5 Aor 84 Specimen Number: // (11th from west, 5th from east JACK GAUGE NET INSERT DEFLECTION (IN.) PRESS. THRUST JACK LOAD NOTES-FAILURE MODE THRUST AFTER (Lb.) 2-MIN (Lb.) (Lb.) INIT. 2650 200 1425 2850 0.000 0,000 8/50 400 5300 4 075 0.000 0.000 6 725 1000 600 .000 7 950 9 375 201 800 ,001 18 750 10 600 13 250 24050 .0035 1000 ,003 14 675 1200 15 900 29 350 34 650 .006 .005 1400 .009 .011 21 200 19 975 39 950 ,015 1600 1800 .0155 .017 25275 26500 2000 .020 50 550 .0195 2200 29 150 27 925 55050 .022 .023 2400 31 800 61 150 .027 .028 33 225 2000 34450 66 950 .032 .035 35 875 2800 37/00 .073 .078 .096 3000 39750 77050 .099 3200 41 175 1055 42400 82 350 103, +00 .1113 3400 45 050 123 43825 87650 46 475 50,350 92 950 3800 .138 53,100 103 550 4000 53,000 .190 Adrupt failure of 54 325 2100 threads (Insert and red). "Was "full" Threads on both insort were stripped. Concrete spalled to short 12 depth, 18 x 15" cover concrete only. Rebar not exposed. By dynamometer * Jack Thrust (Lb.) = Gauge Pressure (PSI) x /3.25 due Apr. 17, 84 Total Weight of Load Beam = 2450 ---** Net Jack Thrust = Total Thrust Minus 1/2 Weight of Beam. (with of bin. = 1225#) ** Insert Load = Net Jack Thrust x 2. Jack:.....Equipment Number RCH 606 Pressure Gauge: M & TE Number 2355 Due Date: 16 Apr 84 Dial Gauge: M & TE Number Due Date: 29 Jun 84 Performed By:, Witnessed By:

tetral

Hilbelf 5 apr 84

Name

COMANCHE PERSONSES TENSION TESTS

RICHMOND 12-INCH, TYPE INSERT

Reference: CP-EI-13.0-\$ 13,0H

Specimen Number: 12 Load Rod Spec: A-193 Date: 5 April 84

SAUGE	JACK_	NET IACK	INSERT	DEFLECTION	ON (IN.)	
RESS.	THRUST (Lb.)	JACK THRUST '(Lb.)	LOAD (Lb.)	INIT.	AFTER 2-MIN.	NOTES-FAILURE MODE
200	2650	1425	2850	0.000	0.000	
400	5300	4075	8150	0.000	,000	
6.00	7950	6725	13 450	,000	1000	
800	10600	9375	18750	.0015	.002	
000	13 250	12 025	24 050	,0035	.0055	
1200	15 900	14675	29 350	.007	,008	
1400	18 550	17 325	34 650	,009	.010	
1600	21 200	19 975	39950	.0115	.012	
1000	23 850	22625	43250	,014	-0145	
2000	26 500	25 275	50550	.017	.0175	
2200	29150	27 925	55000	.0195	.020	
2400	31800	30 575	61150	,022	.0225	
2600	39450	33 225	66 450	.0245	,0265	
2800	37 100	35875	71 750	-028	.0295	
3000	39 750	38525	77 050	,032	,034	
3200	42 000	41 175	82 350	.036	,037	
3400	45 050	43825	87 650	.040	.043	
3600	47 700	46 4.75	92 950	,048	.057	
3800	50 350	49 125	98 250	.057	.0625	
4000	53 000	51 775	103 550	.070	.075	
4200	55 650	54 425	108 850	.084	,092	- 4
4400	58300	57075	114.150	.120		Failure by strippe
			was full.	" stripp	ed lenat	in 18" dig. ares
				apparenty	y result	+ of impact
			abrupt	Max der	th of st	ris failure was
			Was /"	Did not	expose	rebar.

**	Jack Thrust (Lb.) = Gauge Pressure Total Weight of Load Beam = 2450 Net Jack Thrust = Total Thrust Minus Insert Load = Net Jack Thrust x 2. Jack:Equipment Number R	s 1/2 Weight of Beam. ($\frac{1}{2}Wf = 1225$)
	Pressure Gauge: M & TE Number 23	
	Performed By: Och Hilbert 5 apr 84 Name Date	Witnessed By: Advantitude 9-5-84 QA Representative Date

COMANCHE PEAK JES TENSION TESTS

RICHMOND / = INCH, TYPE INSERT

Reference: CP-EI-13.0- 13.0 €

Specimen Number: 13 Load Rod Spec: A-193 Date: 5 Apr 84

GAUGE	JACK	NET NET	INSERT	DEFLECTION	ON (IN.)	
PRESS.	THRUST (Lb.)	JACK THRUST (Lb.)	LOAD (Lb.)	INIT.	AFTER 2-MIN.	NOTES-FAILURE MODE
200	2650	1425	2850	0,000	0,020	
			8/50	0.000	0,000	Programme Company
400	7950	6725	13 450	0,000	0,000	
200	10 600	9375	18 750	0.000	0,000	Art of the colour and the same
1000	13 250	12 025	24 050	0.00/	0,001	
1200	15 900	14 675	29 350	.001	.001	
1400	18 550	17325	39650	.00/5	.00/5	
1600	21 200	19975	39 950	:003	.004	
1800	23850	22 625	+ 3 250	.0045	.0045	
2000	26500	25 275	20 530	10055	,007	
2200	29150	27 925	55 850	.0075	,008	
2400	31 800	30 575	61150	.009	.010	
2600	34 750	33 225	66 450	.011	.012	
2800	37 100	35 875	71 750	R0195	.015	
3000	34 750	38 525	77 050	,0175	.0185	
3200	42400	41 175	82 350	,021	,023	
3400	45 050	4825	87650	,0255	,0285	
3600	47 700	+6475	92 950	,033	,0385	
3800	50350	49 125	48 250	.045	.051	
4000	53 000	51 775	10 3 550	.059	,063	
42.00	55 650	54 425	108 050	.074	.080	
4400	58,300	57.075	114,150 -			Concrete failed
			structura	1 failure	that allo	ned this was
		The Park of		t the m		ecting the
			axial str		to The	threaded coil.
			concret	the state of the same of the s	or then	
			Concret	visible de	rebar o	eath looked into
			tapped v	vith a m	etal ob	ike a void wire

* Jack Thrust (Lb.) = Gauge Pressure (PSI) x 13.25

Total Weight of Load Beam = 2450

** Net Jack Thrust = Total Thrust Minus 1/2 Weight of Beam. (½ Wf. = 1225#)

*** Insert Load = Net Jack Thrust x 2.

Jack:.........Equipment Number RCH 606

Pressure Gauge: M & TE Number 2355 Due Date: 16 Apr 84

Dial Gauge: M & TE Number 2949 Due Date: 29 Jun 84

Performed By:

C Wilbett 5 apr 82

Andrew Retrice 4.5.24

QA Representative Date

COMANCHE PEAK SES TENSION TESTS

RICHMOND 1/2-INCH, TYPE INSERT

Reference: CP-EI-13.0-\$ 13 sey

Specimen Number: 14 Load Rod Spec: A - 193 Date: 5 Apr 84

(14th from West End, 2nd from East) ** JACK NET GAUGE INSERT DEFLECTION (IN.) PRESS. THRUST LOAD JACK NOTES-FAILURE MODE THRUST AFTER (P.S.I.) (Lb.) (Lb.) (Lb.) INIT. 2-MIN 0,000 200 2650 0.000 1425 2 850 4075 400 5300 8 150 0,001 0.001 -0015 7950 6725 600 13 450 ,0011 10 000 18 750 .000 ,002 800 12 025 15 900 29 050 .004 1000 .004 1200 .0065 1004 .0005 1400 18 550 17 325 34 650 .000 27 750 21 200 1600 19 975 ,0095 .0045 22 625 1800 23 850 43250 .010 .010 26 500 2000 ,010 50550 ,010 2200 29150 27 425 55 850 .010 .010 2400 31800 30 575 61 \$50 .012 .012 35 875 2600 71 750 34450 .0135 .014 37 100 2800 .010 .0165 38 525 41 175 43,825 77 050 82 350 87,650 59 750 018 3666 .019 3200 42 400 .024 .020 .055 Concrete failed 3400 ,028 2000 cons type tailure. Depth of cone Shear of insert Top of dooth caual initial failure, rehars lifted Nimited cone After concrete, fracturing an area cover the long of men 3'x5' about ponding to the direction of upper

*	Jack Thrust (Lb.) = Gauge Press Total Weight of Load Beam = 24	ure (PSI) >	(13.25-
***	Net Jack Thrust = Total Thrust Minus 1/2 Weight of Beam. (* ** = 1225*) Insert Load = Net Jack Thrust x 2. Jack: Equipment Number RCH 606		
	Pressure Gauge: M & TE Number_	2355	Due Date: 10 Apr 84
	Ofal Gauge: M & TE Number	2949	Due Date: 29 Jun '84
	Performed By:	,eu	Witnessed By:

Date

Date

Representative

COMANCHE PEAK SES TENSION TESTS

EC-6W RICHMOND /2 - INCH, TYPE __ INSERT

Reference: CP-EI-13.0=13,cH

Date: 4 April 84 Specimen Number: 15 Load Rod Spec: A-193 (15th from West end - 15 on East End) GAUGE JACK NET INSERT DEFLECTION (IN.) PRESS. THRUST LOAD JACK NOTES-FAILURE MODE AFTER THRUST (P.S.I.) (Lb.) (Lb.) (Lb.) INIT. 2-MIN. 1425 2850 200 0,000 0,000 2650 4075 400 8150 0,000 0,000 5300 0,001 6725 300 7950 13 450 18 750 0.00 3 10 000 9 375 0,003 0.006 000 13 250 12 025 24 050 0.004 29 350 1200 14 675 -008 ,008 17 325 39 950 18 550 -000 1400 .009 2/200 .010 .012 1600 23850 1800 22 625 45 250 .013 .015 50550 25275 .019 .0195 2200 27 925 29 150 55850 .021 .024 2400 30575 61150 .026 .027 31 800 2600 34450 66450 .028 .031 7/750 37/00 35875 2800 .034 1036 .040 ,038 39 750 38525 77050 3000 3600 .042 .041 4200 41175 82 350 45050 87 650 .049 2400 43825 .053 47700 92950 46475 2600 ,058 .065 50,450 3800 50 350 ,069 .081 .70 3400 51,675 100,900 concrete failed pattern Concrete rebar out above come below come Spolled Come about 10" dia at top. conc. to offict dimension top of in vertical rods of insert, (Abt. 6")

* Jack Thrust (Lb.) = Gauge Pressure (PSI) x 13.25

Total Weight of Load Beam = 2450 46. (+2 = 1225)
** Net Jack Thrust = Total Thrust Minus 1/2 Weight of Beam. By dynamoten No. M & TE 1432 due Apr 17, 84 *** Insert Load = Net Jack Thrust x 2.

Jack:.... Equipment Number RCH 6C6

Pressure Gauge: M & TE Number 2355

Due Date: 16 Apr 84 Due Date: 29 Jun 84

2949 Dial Gauge: M & TE Number

Witnessea By:

Performed By:

Hame

RICHMOND / -INCH, TYPE INSERT

Reference: CP-EI-13.0-X/2ge#

Specimen	Number: 16	Boit	Spec: 4-193-490	Date: 4	6 april	84
1154	- 141-141		1	_		

DEFLECT	ON (IN.)	GAUGE	JACK *	
INITIAL	AFTER	PRESSURE	THRUST	NOTES-FAILURE MODE
	2-MIN.	(P.S.I.)	(Lbs.)	
0.000	0,000	400	5300	
.001	.001	800	10,600	
.0195	,021	1200	15,900	
.042	.004	1600	21,200	
,062	.0655	2000	26.500	
,085	.091	2400	31,800	
+0112	.123	2800	37,100	
-152	.170	3200	42,400	
.22		3,500-3600	46,375	Failure of both in shear.
		Insert top o	Veflected	1/8" by crushing of upper
		portion of	concrete	Within this yield pattern
		the top of	insert rot	ated a few degrees.
				والقرار م وقاله والمواجود ومحساطوا سنادا
			7. Dament -	

•	Jack Thrust equal Shear Load on Insert. Jack Thrust (Lbs.) = Gauge Pressure (PSI) x	13.25		
	Jack:Equipment Number RCH GOG			
	Pressure Gauge: M & TE Number 2355	Due Date: 16	Apr	240
	Dial Gauge: M & TE Number 2749	Due Date: 29		

Performed By:

Jame Hillfolk, 6 april 84

Adem active 4-0-94

Representative - Dat

EC-2W RICHMOND / -INCH, TYPE INSERT

Reference: CP-EI-13.0-X 13 gc # # A-490 Bolt Spec: A-490 gc 4 Date: 6 Apr 84 Specimen Number: 17

2nd fr West End)

DEFLECTIO	ON (IN.)	Vack GAUGE	Gauge *	
INITIAL	AFTER 2-MIN.	Thrust (Lbs)	Pressure (PSI)LOS.)	NOTES-FAILURE MODE
0.000	0,000	5300	400	
.020	.020	10,600	800	
.037	.039	15,900	1200	
.060	.0645	21200	1600	
,087	,093	26 500	2000	
,127	.129	3/ 800	2400	
.166	-186	37/00	2800	
.3/3	. 332	42400	3200	
		43 060	3250	Failure by bolt shear
		Inse	+ deflecte	d horizontally 3/8, being
		hero	nitted bu ch	ushing failure of concrete
		No e	apparent re	usting failure of concrete
		he in place.		

Jack Thrust equal Shear Load on Insert. Jack Thrust (Lbs.) = Gauge Pressure (PSI) x	13.25	
Pressure Gauge: M & TE Number 2355 Dial Gauge: M & TE Number 2949	Due Date: 16 Due Date: 29	Apr 84

Performed By:

C Hilleth 6 apr 84

RICHMOND / -INCH, TYPE INSERT

Reference: CP-EI-13.0-X 420 W

Specimen Number: 18 Bolt Spec: A - 490 Date: 6 Apr 84

(3rd from west end)

DEFLECTION (IN.)		GAUGE	JACK *	
INITIAL	AFTER 2-MIN.	PRESSURE (P.S.I.)	THRUST (Lbs)	NOTES-FAILURE MODE
	C-112111			
0.000	0,000	400	5-300	
.003	.004	800	10 600	
.02	.0245	1200	15 900	
.042	.045	1600	21 200	
.060	,063	2000	26,500	
.080	,085	2400	31 800	
.104	.109	2800	37100	
.136	.148	3200	42400	
.200	.332	3600	47 700	
.400		3800	50.350	Failure by bolt shear.
				Insert top deflected about 1/8", no
				apparent rotation of insert.
				Top of concrete crushed (spaller
				about 2" in front of insert.
Mineral				The insert washer sheared off
			Continue to	from the strute, thus the "
				deflection was after this shear
			I property	failure Coils & struts did
				not move.

*	Jack Thrust equal Shear Load on Insert. Jack Thrust (Lbs.) = Gauge Pressure (PSI) x	13.25	
	Jack:Equipment Number RCH 606 Pressure Gauge: M & TE Number 2355	Due Date: 16 Apr	84
	Dial Gauge: M & TE Number 2949	Due Date: 29 Jun	

Performed By:

Al C. Gilliott, 6 apr 84

Ada, Ricking 4-6-84

A Representative Date

RICHMOND / - INCH, TYPE INSERT

Reference: CP-EI-13.0-X 324

Specimen Number: 19 Boit Spec: 4-490 Date: 9 Apr 82

(4th from west end)

DEFLECTI	ION (IN.)	GAUGE	JACK *	
INITIAL	AFTER 2-MIN.	PRESSURE (P.S.I.)	THRUST (Lbs.)	NOTES-FAILURE MODE
0.004	0.0035	400	5300	
.036	.036	800	10 600	
.052	.0605	1200	15 900	
,080	,081	1600	21200	
.098	.099	2000	26 500	
.122	./27	2400	3/800	
.147	.155	2800	37 100	
.190	.2225	3200	+2400	
		3:00	E BENEFIC STORY	
, 270	THE PERSON NAMED IN	3500	46,375	Insert failed by breaking
				weld between upper coil and struts, Boit failed after rotating
		TAR SHOWS US		with the engaged upper coil
				thru several degrees. The
	Section 1			bolt failed in bending with
10 210 71			BULL TENEDLEN	a lesser load than the
	CARRIED IN			46 375 16.
	Note that the same			
			The state of the s	

Jack Thrust equal Shear Load on Insert. Jack Thrust (Lbs.) = Gauge Pressure (PSI) x	13.25		
Pressure Gauge: M & TE Number 2355 Dial Gauge: M & TE Number 2949	Due Date:_ Due Date:_		
Performed By:	Witnessed	Ву:	

Performed By:

1 C. Hillith 9 apri 84

EC.2W

RICHMOND / -INCH, TYPE INSERT

Reference: CP-EI-13.0-X/39c4

Specimen Number: 20 Bolt Spec: A-490 Date: 9 apr 84

(5th from West End)

DEFLECTI	ON (IN.)	GAUGE	JACK *	manife the same
INITIAL	AFTER 2-MIN.	PRESSURE (P.S.I.)	(Lbs.)	NOTES-FAILURE MODE
0.007	-0,0:7	400		
	Kelenakan l	-800		
0,000	0.00	400		Slack not out of apparatus
1008	1008	800		
	6492206	1200		
0.003	0,003	900	5300	
.025	.032	800	10600	
.046	.096	1200	15900	
,063	.06,0	1600	2/200	
.085	.087	2000	26500	
.115	A:122	2400	31800	
.154	. 173	2800	37100	
,270		3200 3000	39,750	Concrete crushed insert
				remained intact but upper
				portion rotated thru a few
		CHECK LINES WAS		degrees. Deflection of upper
				part of insert (washer) 3/8:
				Bolt broke in bending at
				lower load than the max
				39.750, Rotation caused come spa
			+ +0 3	to lift on tension side 1/2 occo typerin

Jack Thrust equal Shear Load on Insert.

Jack Thrust (Lbs.) = Gauge Pressure (PSI) x

Jack:.....Equipment Number RCH 606

Pressure Gauge: M & TE Number 2355

Dial Gauge:....M & TE Number 2949

Due Date:

Performed By:

QC Hilbeth 9 apr'84

to sero 10" back. Spall total 12"die (I)

Due Date: 16 apr 84
Due Date: 29 Jun '84

Witnessed By:

Ale Chiefine 1-9-84

QA Representative Date

COMANCHE PEAK SES COMBINED SHEAR & TENSION TESTS

Richmond / -Inch. Type Insert
Reference: CP-E1-13.0-\$/37.4

Common	SHEAR		MINE N			TENSI	ON			
Gauge Press. (PSI)	Jack Thrusk (Lb.)		ction ch) After 2-Min.	Galige Press. (PSI)	Jack Thrust (Lb.)	het Mck Thoust (Lp.)	Injert Load (Lo.)	and the same of the same	ection och) After 2-Min.	Notes - Failure Mode
400	5300	0.000	0.000	1	5,300	3		0.000	0.000	
800	10 600	.001	.001	(10,600)	.000	.000	
1200	15900	.065	.061	_ <	15,900			.022	. 0225	
1600	21200	.173	-/83	_ <	21,200	7		.049	. 052	
2000	26500	, 330	.378		26,100			.111	.134	
240000	27,825	.400		}	27.825	1	3	.15		Sound of a weld breaking. Rod sheared
				*/ _A		M/A	*/A			e esig. = 900 × 11,925 the linear washer deflected laterally 30". Some bending of red but tractured surface indicated a stream break. Top of insert rotated thru Concrete spalled all around @ about 12" dia. 12" deep @ insert, pero depth @ edge

1-* Jack Thrust (Lb.) = Gauge Pressure (PSI) x 13.25 for Shear Load.

2-* Jack Thrust (Lb.) = Gauge Pressure (PSI) x 13.25 for Tension Load.

Total Wt. of Tension Load Beam = N/A 1b.

** Not Jack Thrust = Total Thrust Winus 1/2 Wt. of Beam.

** Insert Load = Net Jack Thrust & 2.

Pressure Gauge-M&TE No: 2355 Due Date: 16 am 84 Dial Gauge-M&TE No: 2849 Due Date: 28 Jun 84

Tension Apparatus: Jack-Equipment No: 2949 Due Date: 29 Jun 84

Tension Apparatus: Jack-Equipment No: RCH 603T

Pressure Gauge-M&TE No: 2950 Due Date:

Dial Gauge-M&TE No: 2950 DPg/Due Date: 18 Jun 84

COMANCHE PEAK SES COMBINED SHEAR & TENSION TESTS
Richmond / -Inch. Type Insert

Reference: CP-EI-13.0-#/9,cw

Specimen Number: 22 (7th from West) Inserted Load Rod: A-193 Date: 9 April 84 Comman TENSION 1-* Gauge Jack Deflection Gauge Press. Net Jack Insert Deflection Press. Thrust (Inch) Thrust Jack Loud (Inch) Notes - Failure Mode After Turust After (Lb ' (PS1) Init. 2-Hin. (P\$1) (Lb.) (Lh.) (D).) Init. 2-Min. 400 0,000 0.004 5 300 0.001 0.00/ 800 10,600 .037 .038 10 600 013 .014 1200 15,900 .105 .109 15 900 .0255 .026 1600 .196 .205 21,200 21 200 055 .058 2000 26,500 . 342 .428 26500 .115 .145 2200 29 150 .52 28150 .16 25 850 Weter droke sharp naise. Couse unknown 1800 23 850 1500 19875 19875 Rod sheared. Kod had rotated e shear line thru approx, 20 when broke. Concrete spelled approx 15" diameter, being 12" on tension side & 3" on comp. (town shear jack) side. 2" deep & insert. Insert remained intact.

1-* Jack Thrust = Shear Load on Insert.

1-* Jack Thrust (Lb.) = Gauge Pressure (PSI) x 19.25 for Shear Load.
2-* Jack Thrust (Lb.) = Gauge Pressure (PSI) x 13.25 for Tension Load.
Total Wt. of Tension Load Beam = N/A Lb.

** Not Jack Thrust = Iotal Thrust Hinus 1/2 Ut. of Beam.
Insert Load = Net Jack Thrust x 2.

Performed By: Q. C. Hillsty 9 man 84.

Shear Apparatus: Jack --- Equipment No: RCH GO6

Pressure Gauge-M&TE No: 2955 Due Date: 16 am 14 Dial Gauge-M&TE No: 2949 Due Date: 29 Jun 89

Tension Apparatus: Jack-Equipment No: RCH 6037

Pressure Gauge-M&TE No: Jame Due Date:

Dial Gauge-M&TE No: 2094 Due Date: 18 Jun 84

Witnessed By: Ale Critical 4-9-14

OA Representative Date

COMANCHE PEAK SES COMBINED SHEAR & TENSION TESTS Richmond / -Inch, Type Insert Reference: CP-E1-13.0-FpdH

Date: # apr 84 Specimen Number: 23 (8th from west and) Inserted Load Rod: A-193 SHEAR Common **TENSION** 1-* Gauge Jack Deflection Gauge Jack Net Deflection Insert Press. Thrust (Inch) Press. Thrust Jack Load (Inch) Notes - Failure Mode After Thrust After (PS1) (Lb.) 2-Hin. (PSI) (Lb.) (Lb.) (Lb.) Init. 2-Min. 400 5300 0.002 0.006 5 300 0.0005 0.0005 800 10 600 035 ,0375 10 600 009 010 1200 15 900 .122 15 900 .134 038 ,041 1600 21 200 .290 ,269 21 200 075 .084 2000 26500 .350 26 500 .410 158 .140 2200 29 150 .430 29150 Deflection increased rapidly. 20 NA 2300 30 475 .620 30475 N/4 Abrupt failure by shear of \$ rod. Insert washer moved horizontally 1/9". No breakage of insert, Rod rotated some 30° above threads of insert. This permitted by crushing of concrete and probably defor mation of threaded coil. Rod failure was by shear after considerably deformation

1-* Jack Thrust = Shear Load on Insert.

1-* Jack Thrust (Lb.) = Gauge Pressure (PS1) x /3.25 for Shear Load.

Jack Thrust (Lb.) = Gauge Pressure (PSI) x 13.25 for Tension Load.

Total Wt. of Tension Load Beam = N/A lb.

At Het Jack Ihrust = Total Thrust Hinus 1/2 Ht. of Beam.

*** Insert Load = Wet Jack Thrust

Performed By: Q. C. Hillial 10 Ger 84

Shear Apparatus: Jack --- Equipment No: ACH GOG

Pressure Gauge-M&TE No: 2355 Due Date: KG Aur 84
Dial Gauge-M&TE No: 2949 Due Date: L9 Jun 84

Tension Apparatus: Jack-Equipment No: 8CH 603T

Pressure Gauge-M&TE No: 2094 Due Date:
Dial Gauge-M&TE No: 2094 Due Date: 18 Jun 84

Witnessed By: Adres With 4-13-84

COMANCHE PEAK SES COME INED SHEAR & TENSION TESTS Richmond / -Inch, Type Insert Reference: CP-EI-13.0-5904

Specimen Number: 24 (9th from west end) Inserted Load Rod: 4-193

Common	SHEA	R				TENSI	ON										
Gauge Press. (PSI)	1-* Jack Thrust (Lb.)		-(1)	-(1)	-(1)	-(1)	-(1)	-(1)	Init.	ection nch) After 2-Min.	Gauge Press (PSI)	. Thrust	Net Jack Thrust (Lb.)	Insert Load (Lb.)	(In	ction ch) After 2-Min.	Notes - Failure Mode
400	5,300	0.001	0. 001		5,300	1	1	9.002	9002								
800	10,600	,008	100.		10600	3	5	.0065	.0065								
1200	15 900	,060	.070	1	15 900	7	5	.027	.030								
1600	21 200	.153	.171	5	2/ 200	3	>	.062	.069								
2000	26 500	, 325	,390	5	26 500	5	>	-135	.159								
2100	37 826 37 800	,400			27 825	3	3	.17		Rapid yielding began.							
2100	34 80a	.500		N/A	27825	N/A	1/4	,20		July Vigon							
2200	48 450	.540			29150	1	1	.227									
	29150	.700	Parent.	1	29150	3	3	.227									
2000	26500			1	26500	5	5			Bueak. Abrupt shear failure of rod.							
				5			_			Some 1/8" norigontal deflection of top							
				1		5	}			of inert permitted by totalion crushing							
						-	_ {			of concrete and deformation of upper							
				5		2	2			coil of insert. Concrete cost 12" done							
							Sten	vian. V.	116/ 29	of insert seen to have tilted 5 (1). Insert Inta							

1-* Jack Thrust = Shear Load on Insert.

1-* Jack Thrust (Lb.) = Gauge Pressure (PSI) x /3.25 for Shear Load.
2-* Jack Thrust (Lb.) = Gauge Pressure (PSI) x /3.25 for Tension Load.
Total Wt. of Tension Load Beam = //4 Lb.
** Not Jack Thrust = Total Thrust Minus 1/2 Wt. of Boam
*** Insert Load = *Net Jack Thrust x 2...

Shear Apparatus: Jack---Equipment No: RCH COG

Pressure Gauge-M&TE No: 2355 Due Date: 16 and by
Dial Gauge-M&TE No: 2949 Due Date: 29 for 85

Tension Apparatus: Jack-Equipment No: RCH COST

Pressure Gauge-M&TE No: dame Due Date:
Dial Gauge-M&TE No: 2094 Due Date: 18 don 84

Performed By: Q. C. Hilberth, 10 april 84

COMANCHE PEAK SES COMBINED SHEAR & TENSION TESTS Richmond / -Inch, Type Insert Reference: CP-E1-13.0-FPEN

Specimen Number: 25 (10th from West and) Inserted Load Rod: 4-193 Date: 10 April 84 SHEAR Common TENSION 1-* 2-* Gauge Jack Deflection Gauge Jack Net Insert Deflection Press. (Inch) Thrust Press. Thrust Jack Load (Inch) Notes - Failure Mode After Thrust After (PSI) (Lb.) Init. 2-Hin. (PSI) (Lb.) (Lb.) (Lb.) Init. 2-Min. 400 5 900 0.0015 0.0015 0,000 5300 0,000 800 10 600 0.026 10 600 .028 0025 .004 1200 15 900 .115 .122 15 900 -1005 21 200 1600 .217 ,229 21 200 .010 .006 2000 26 500 . 260 .324 26 500 1049 .063 2100 27825 ,320 27825 .070 NA 28 987 NA 2150 .400 28.487 Abrupt break, Insert deflected 1/2"(-) .080 Rod failed in shear. 1-* Jack Thrust = Shear Load on Insert.

1-* Jack Thrust (Lb.) = Gauge Pressure (PSI) x 19.25 for Shear Load.
2-* Jack Thrust (Lb.) = Gauge Pressure (PSI) x 13.25 for Tension Load.
-- Joel Wt. of Tension Load Beam = N/A Lb.
** Net Jack Thrust = Total Thrust Hinus 172 Wt. of Beam.
** Insert Load = Net Jack Thrust + 2.

Performed By: Q & Gilbeth 10 april 84

Shear Apparatus: Jack --- Equipment No: RCH GOG

Pressure Gauge-M&TE No: 2355 Due Date: 16 an 80 Dial Gauge-M&TE No: 2949 Due Date: 29 Jun 80 Tension Apparatus: Jack-Equipment No: RCH GOST

Pressure Gauge-M&TE No: Jame Due Date: Dial Gauge-M&TE No: 2094 Due Date: 18 Jun 80

Witnessed By: Ala Silly &

COMMICHE PEAK SES TELISION TESTS

EC-2W RICHMOND / -INCH, TYPE INSERT

Reference: CP-EI-13.0- 139cu

Load Rod Spec: A-193 Date: 6 Apr'84 Specimen Number: 26 11th at from west and, 5th from est

GAUGE PRESS.	JACK	NET INCK	INSERT	DEFLECTI	ON (IN.)	
P.S.I.)	THRUST (Lb.)	JACK THRUST (Lb.)	LOAD (Lb.)	INIT.	AFTER 2-MIN.	NOTES-FAILURE MODE
200	2650	1425	2850	0.000	0,000	
400	5300	4075	8150	.003	.003	
600	7 950	6725	13450	.007	.0075	
800	10 600	9375	18750	.012	.0/25	
1000	13 250	12 025	24050	.0175	.019	
1200	15 900	14 675	29350	.037	.038	
1400	18550	17325	34650	.070	.070	
1600	21200	19975	39 950	.098	.105	
1700	22 525	21,300	42,600	.134		Failure.
			insert re	mained	intact.	Shear cone type
			failure o	f concr	ete. Inse	ert was located
			near cen	ter between	reen E-W	e N-s rebars.
			cone was	restric	ted some	what by 4-bers
			2- each w	y. Some	lifting fo	rce onbars caused
			concrete	to so	11 3-14	es, side of insert
			Shear co	ne den	th = full	neight of inser
			less 3/4"	@ botto	m.	

* Jack Thrust (Lb.) = Gauge Pressure (PSI) x /3.25 Total Weight of Load Beam = 2450 ** Net Jack Thrust = Total Thrust Minus 1/2 Weight of Beam. (Wet. = 1225 16) *** Insert Load = Net Jack Thrust x 2.

Jack:.... Equipment Number RCH 606

Pressure Gauge: M & TE Number 2355 Due Date: 16 Apr 84

Due Date: 29 Jun 84

Dial Gauge: M & TE Number

Witnessed By:

Performed By:

COMANCHE PEAK SES TENSION TESTS

RICHMOND / -INCH, TYPE INSERT

Specimen Number: 27 Load Rod Spec: A-193 Date: 6 Apr 84

Reference: CP-EI-13.0-\$139ey

GAUGE PRESS.	JACK THRUST	NET INCK	INSERT	DEFLECT	ION (IN.)	
.S.I.)	(Lb.)	JACK THRUST (Lb.)	LOAD (Lb.)	INIT.	AFTER 2-MIN.	NOTES-FAILURE MODI
200	2650	1425	2850	0.000	0.000	7
400	5 300	4075	8150	.000	,000	for the second second
600	7 450	6725	13450	.000	.000	
800	10600	9375	18750	.0005	,0005	le terrole seglection -
1000	13 250	12 025	24 050	.0065	.0075	
1200	15 900	14 675	29350	.0/65	.0175	
1400	18 550	17 325	34 650	.050	.056	
1600	21200	19975	39 950	1,089,060	.098	International value of the second
2000 17	23,188	21,960	43,920	.146	Market.	Failure
A. Carlo		Biologica Control	Failure	occurered	by failur	e of the insert
			Weld be	Aveen low	er coil	and vertical
		100000000000000000000000000000000000000	struts	broke. Th	V-Baca, U	open, coil canne
		Market St.	out and	carried	the two	strute w/it.
			Concre	te spalle		al area about
			1.5' X 2	25 max		@ insert.
			EXPOSE	d one r	char loc	ated 3" o.c. tr
			insert.			rbed. Only con
			crete d	cover re	moved.	
		-				
			1			
					E HEE	
		-				
	+	 		 		
** Net	Jack Thrust rt Load = Ne	= Total Thru t Jack Thrus	st Minus 1/2	x <u>/3.23</u> Weight of Be	eam. (½ wt.	= 1225)
			er 2355		: 16 Apr	84
Utal	Gauge: M &	IE Number_	2949	. Due Date	: 29 Jun	.84
Perf	ormed By:			Witnesse	ed By:	
1	C. Hills	1 11		1	1.	2 4-6-84
(/	10 40 011	. +4 10	- '01	11 1	/ 1. 1	1 11

COMANCHE PEAK TENSION TESTS

EC-2W RICHMOND / -INCH, TYPE INSERT

Reference: CP-EI-13.0- 13 2CH

Specimen Number: 28

Load Rod Spec: A-193 Date: 10 April 84

(3101 from cost end)

GAUGE	JACK	** NET	*** INSERT	DEFLECT	ION (IN.)	
PRESS.	THRUST (Lb.)	JACK THRUST (Lb.)	LOAD (Lb.)	INIT.	AFTER 2-MIN.	NOTES-FAILURE MODE
200	2650	1425	2850	0,000	0,000	
400	5300	4 075	8150	:000	,000	
600	7 950	6725	13450	,000	,000	
800	10 600	9 375	18 750	.002	,002	
1000	13 250	12,025	24,050	.004	,005	
1200	15,900	14,675	29,350	.009	.010	
1400	18550	17325	34 650	.015	.029	
1550	20 538	19313	38 424	.055		
1600	21,200	19,975	39 950	.067	.082	
1900	22,525	2/300	42,600	.15		concrete shear
2200					Failure.	Insert and rod
2000						Cone height
						it. Size of cone
						er - Hom mot.
				Rebars 1	Hed with	come and lifted
				area 45	× 3.5'.	Rebars @ 9" O.C. E.W

* Jack Thrust (Lb.) = Gauge Pressure (PSI) x /3.25 Total Weight of Load Beam = 2450 ** Net Jack Thrust = Total Thrust Minus 1/2 Weight of Beam. (1/2 Wf. = 1225 Lb) *** Insert Load = Net Jack Thrust x 2. Jack:.... Equipment Number RCH 606 Due Date: /6 Apr 84 Pressure Gauge: M & TE Number 2355 Due Date: 18 wom 84 Dial Gauge: M & TE Number 2049

Performed By:

COMANCHE PEAK SES TENSION TESTS

EC-2W

RICHMOND / -INCH, TYPE INSERT

Reference: CP-EI-13.0--13

Specimen Number: 29 Load Rod Spec: A - 193 Date: 6 April 89

GAUGE PRESS.	JACK THRUST	NET JACK	*** INSERT	DEFLECTI	ON (IN.)	
.S.I.)	(Lb.)	THRUST (Lb.)	(Lb.)	INIT.	AFTER 2-MIN.	NOTES-FAILURE MODE
200	2650	1425	2850	0.000	0.000	
400	7750	6 725	8 150	.009	.005	
800	10 600	9375	18 750	10145	,015	
1000	13250	12 025	24 050	.021	.022	
1200	15900	14 675	29 350	.033	.037	Notice and the second second
1400	18550	17325	\$ 34 650	-101	.104	
1600	21 200	19975	39950	. 135	.1455	
7000		-				
1700	22,525	21,300	42,600		carda fa	iled by the load
		2,000	7-,000		0, 6,0	. 160 -9
	1 - 3		on insel	of lifting	the re	bar mat. An
			then	me 3.5' x insert	Top red	har mat. An led in this mann taking a small faced
			then con	insert interest will tact will	to insert	har mat. An led in this mann to taking a small small placed thus contributed
			then con	insert inthe tact win	to insert	har mat. An led in this mann to taking a small placed thus contribu
			then con	insert interest will tact will	to insert	har mat. An led in this mann to taking a small small faced thus contributed
			then con	insert interest will tact will	to insert	ted in this mann
			then con	insert interest will tact will	to insert	har mat. An led in this mann to taking a small small faced thus contributed
			then con	insert interest will tact will	to insert	har mat. An led in this mann to taking a small small faced thus contributed
			then con	insert interest will tact will	to insert	har mat. An led in this mann to taking a small small placed thus contributed
			then con	insert interest will tact will	to insert	har mat. An led in this mann to taking a small small placed thus contributed
			then con	insert interest will tact will	to insert	har mat. An led in this mann to taking a small small faced thus contributed
			then con	insert interest will tact will	to insert	har mat. An led in this mann to taking a small small faced thus contributed
			then con	insert interest will tact will	to insert	har mat. An led in this mann to taking a small small placed thus contributed
			then con	insert interest will tact will	to insert	har mat. An led in this mann to taking a small small placed thus contributed

*	Jack Thrust (Lb.) = Gauge Pressure (PSI) Total Weight of Load Beam = 2450 Net Jack Thrust = Total Thrust Minus 1/2	
***	Insert Load = Net Jack Thrust x 2. Jack:Equipment Number RCH 64	
	Pressure Gauge: M & TE Number 2355	Due Date: 16 am 84
	Dial Gauge: M & TE Number 2949	Due Date: 29 / 84
	Performed By:	Witnessed By:
	QC Gulbett 6 april 84	Adrew Chitrale 4.5.
	Name Date	QA Representative Date

COMANCHE PEAK SES TENSION TESTS

EC-2W

RICHMOND / -INCH, TYPE ___INSERT

Reference: CP-EI-13.0-Facu

GAUGE PRESS.	JACK THRUST	NET JACK	*** INSERT	DEFLECTION	ON (IN.)	er China Chara (Net serie)
.S.I.)	(Lb.)	THRUST (Lb.)	(Lb.)	INIT.	AFTER 2-MIN.	NOTES-FAILURE MODE
-200				oreca	-0,000	
400				300:	.002	
200	2650	1	2850	0.000	0.000	
400	5300	4075	8 150	0.000	.000	
600	7 950	6725	13450	100/	.00/	المدانية المناسبات
800	10 600	9375	18 750	.005	.006	
1000	13 250	12025	24050	.019	.021	
1200	15 900	14675	29 350		.049	
1400	18 550	17 325	34650	0.106-047	.109	
1600	21 200	19975	39 950	.153	.174	
1650	21860	20635	41270	. 250		Load Peaked Ou
				out w/roo	per (three s, also st	and vertical aged coil) came fruts came out.
				out w/roo Top con surface	per (three street spall only	ruts came out. Dalled abt. 18" dia Called of cover. Bar
				out w/red Top come surface expased.	per (three street spall only	ruts came out. Dalled abt. 18" dia Called of cover. Bar
				out w/red Top come surface expased.	per (three street spall only	ruts came out. valled abt 18" dia val of cover. Bar
				out w/red Top come surface expased.	per (three street spall only	ruts came out. Dalled abt. 18" dia Called of cover. Bar
				out w/red Top come surface expased.	per (three street spall only	ruts came out. Dalled att. 18" dia Called of cover. Bar

Witnessed By:

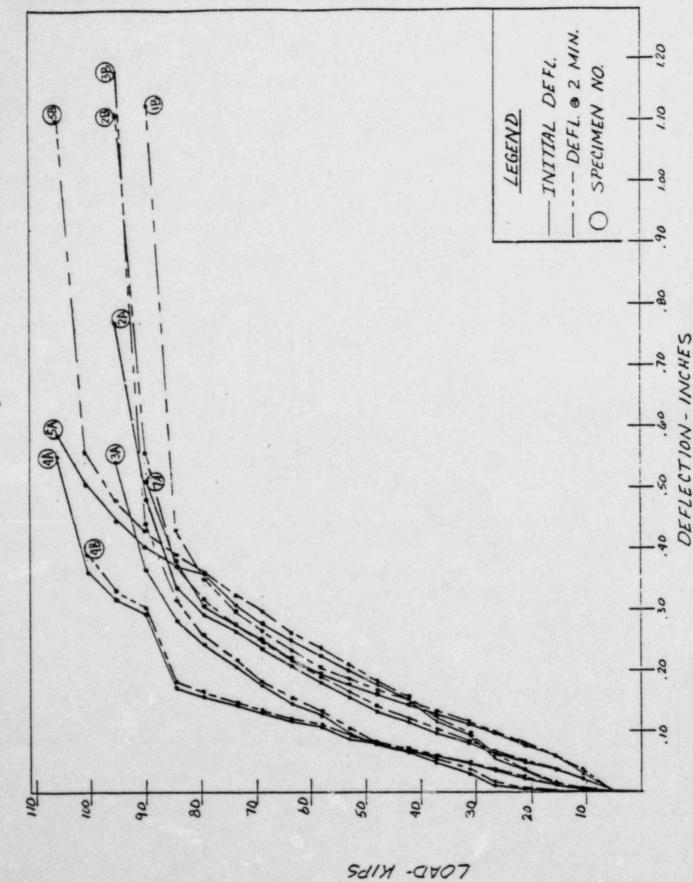
Performed By:

C Hilbert 5 apr 84

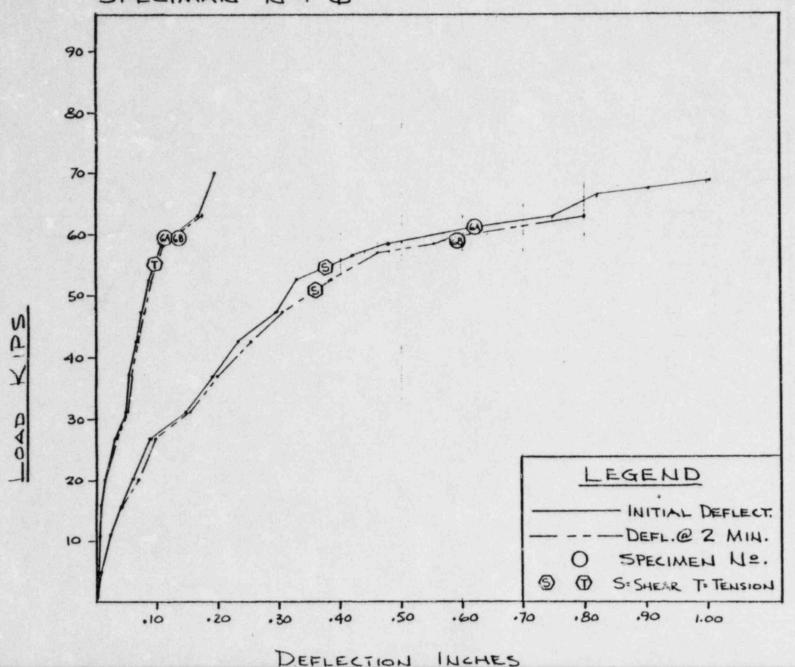
APPENDIX 3

LOAD-DEFLECTION CURVES

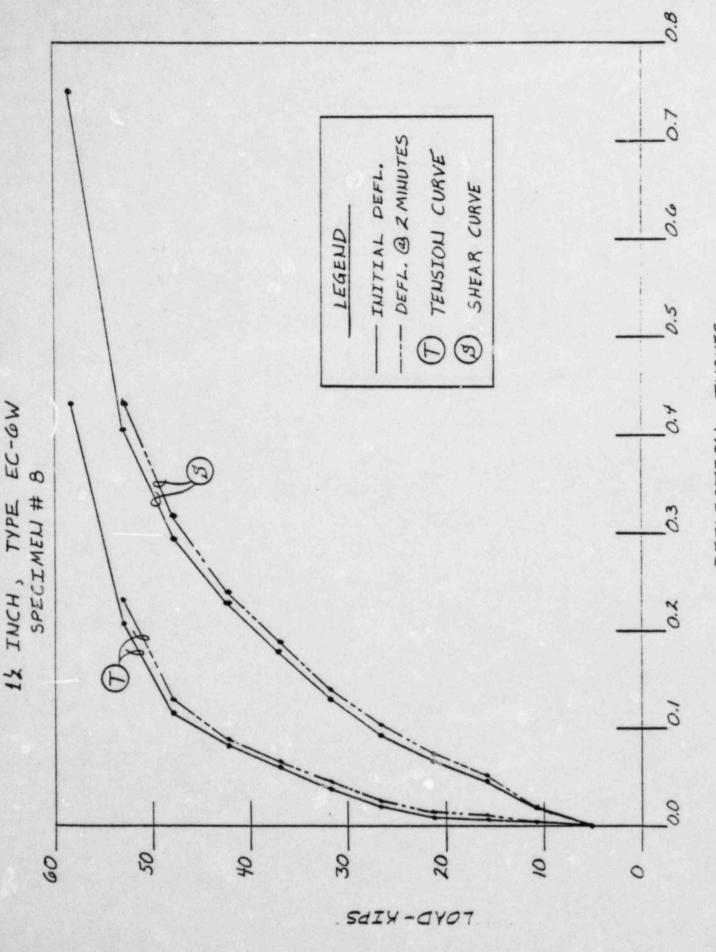
LOAD- DEFLECTION CURVES 11/2-INCH TYPE EC-6W, SHEAR TEST



COMBINED SHEAR & TENSION TEST CHART
RICHMOND 1/2 INCH, TYPE EC-GW INSERT
SPECIMAN Nº. 6

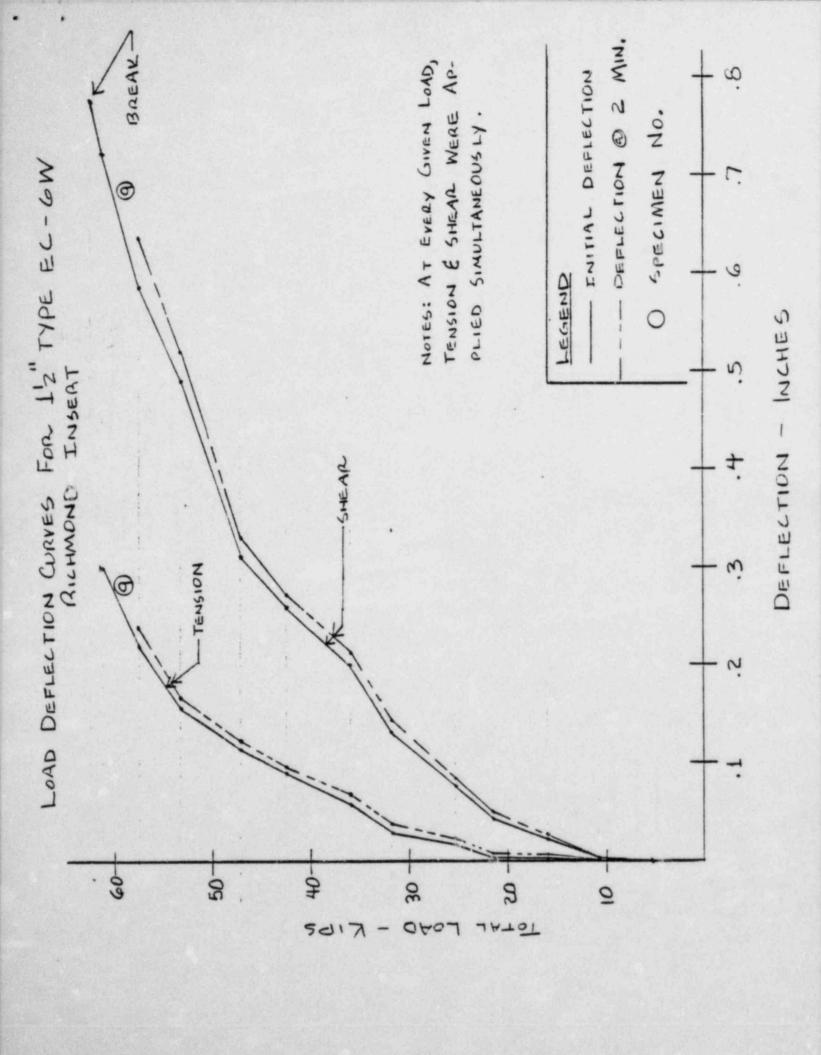


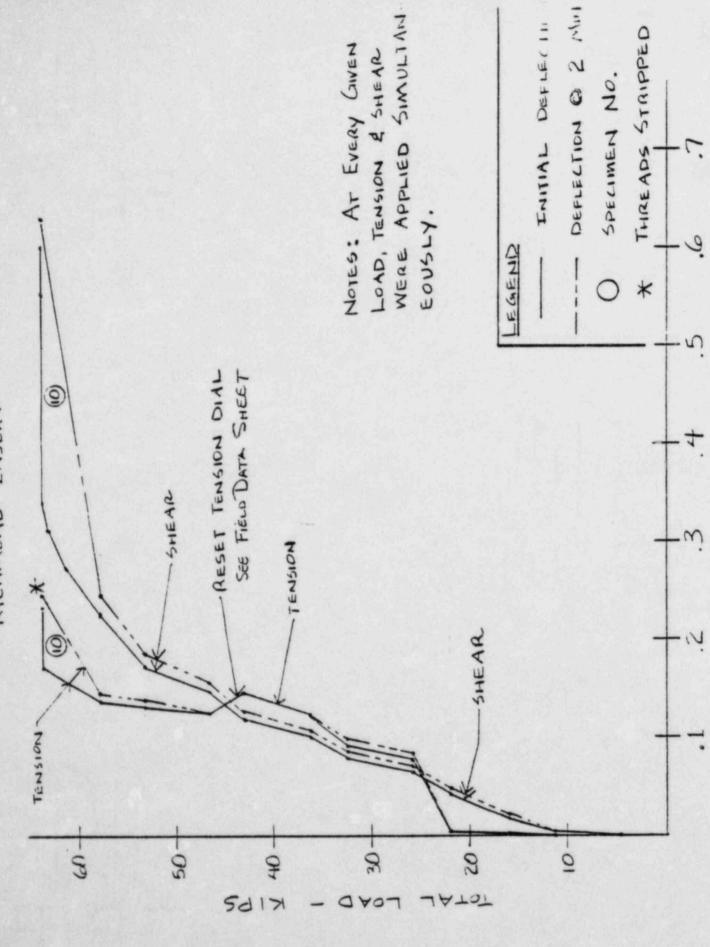
DEFLECTION - INCHES



COMBINED SHEAR & TEMSION TEST CURVES

DEFLECTION - INCHES



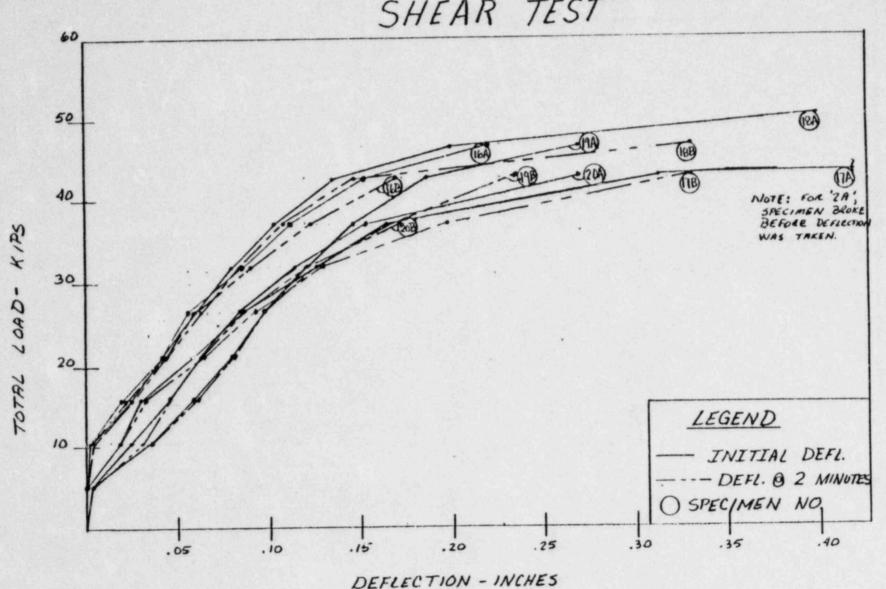


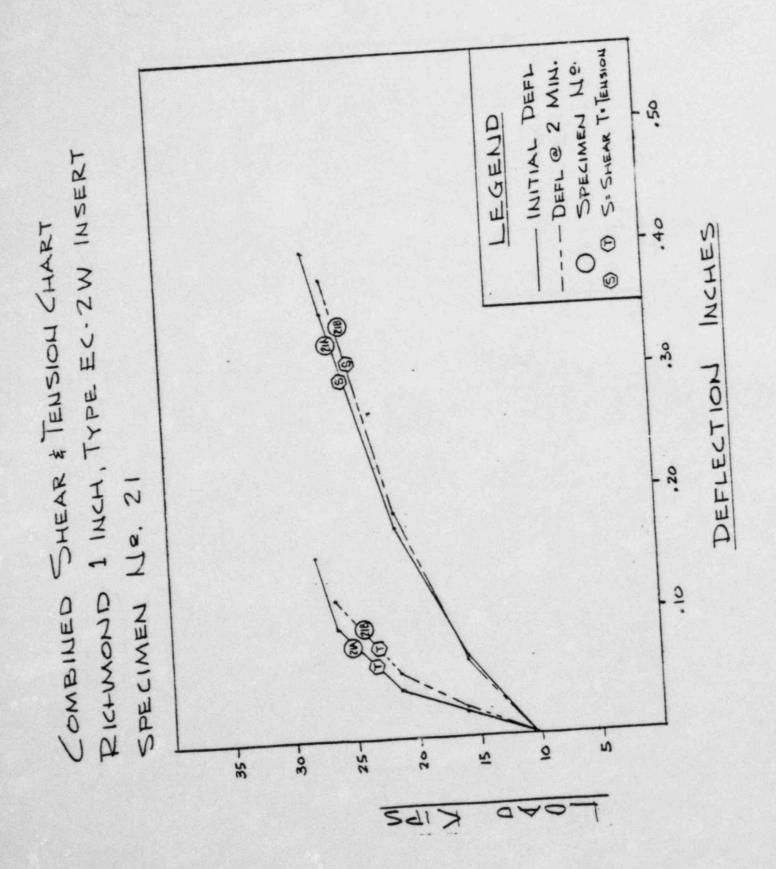
DEFLECTION - INCHES

E E INITIAL DEPLECTION - DEFL. @ 2 MIN .20 O SPECIMEN NO. 12-1NCH TYPE EC-6W, TENSION TEST By C LEGEND DEFLECTION - INCHES 3 8 09 2 120 1/0 100 90

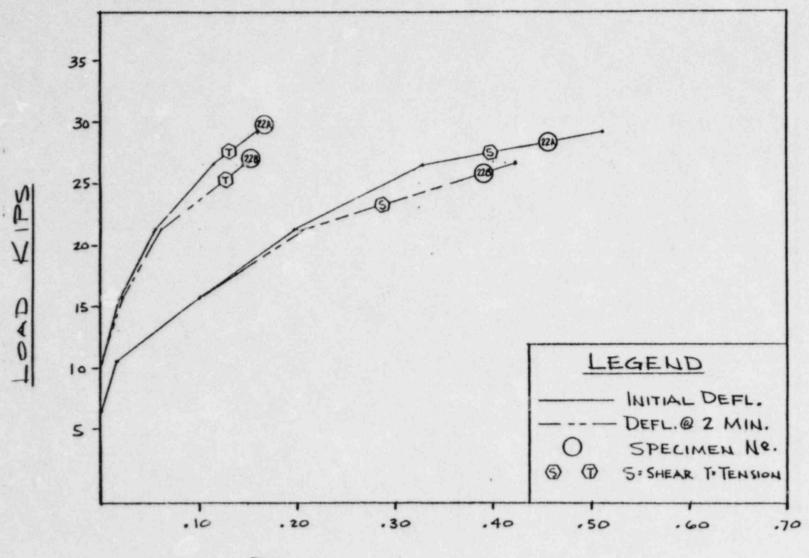
2407

LOAD - DEFLECTION CURVES 1-INCH TYPE EC-2W SHEAR TEST



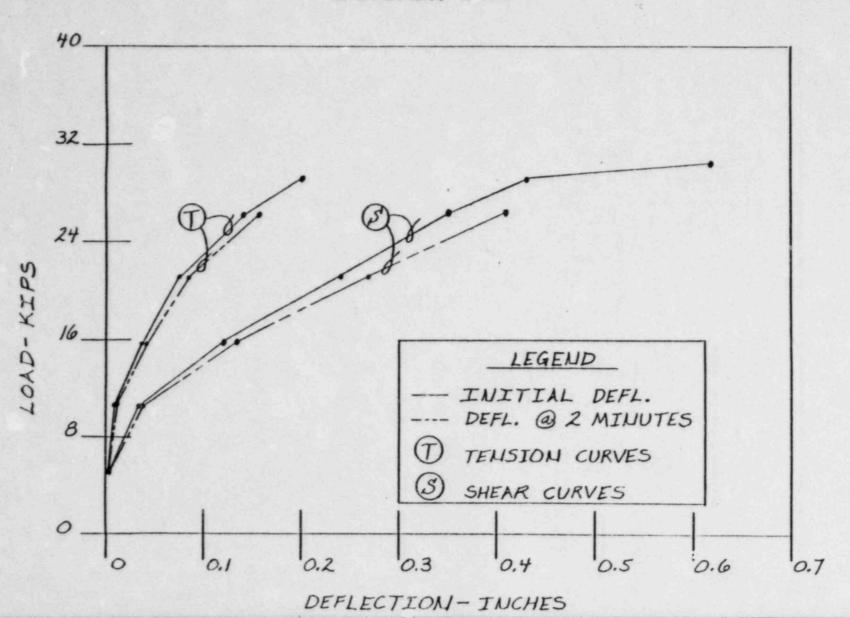


COMBINED SHEAR & TENSION CHART RICHMOND 1 INCH, TYPE EC- 2W INSERT SPECIMAN No. 22

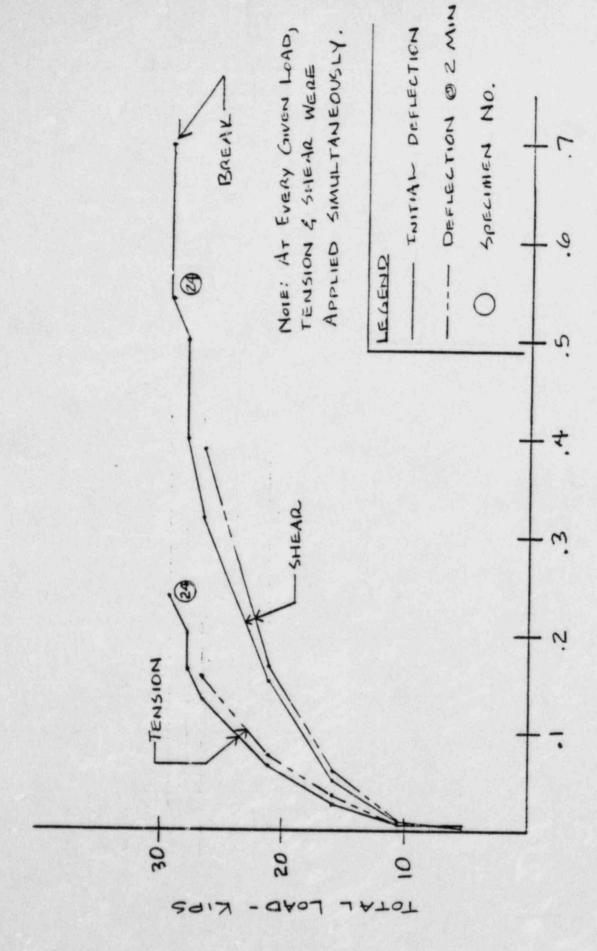


DEFLECTION INCHES

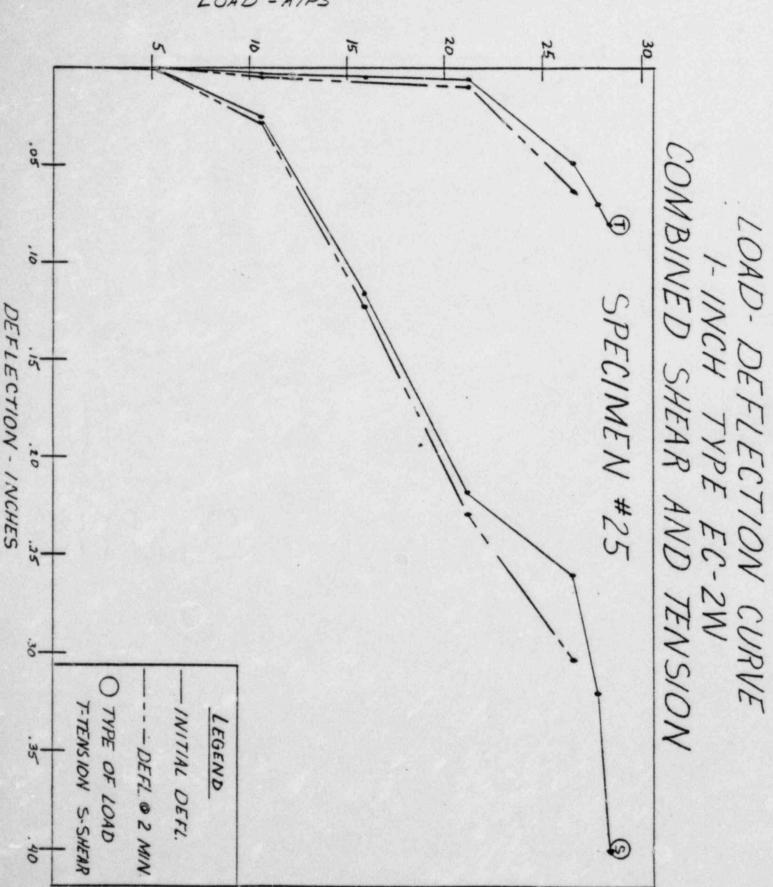
COMBINED SHEAR & TENSION TEST CURVES I INCH, TYPE EC-2W SPECIMEN #23



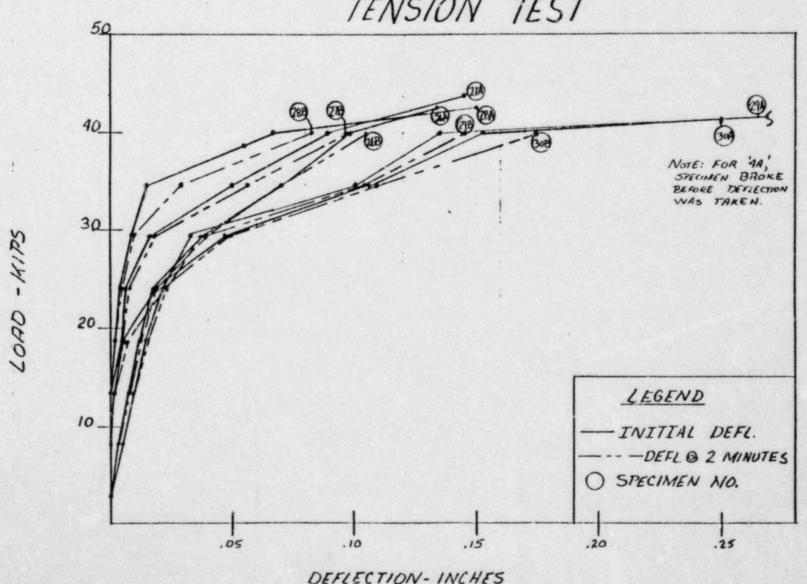
LOAD DEFLECTION CURVES FOR 1" TYPE EC-2W RICHMOND INSERT



DEFLECTION - INCHES



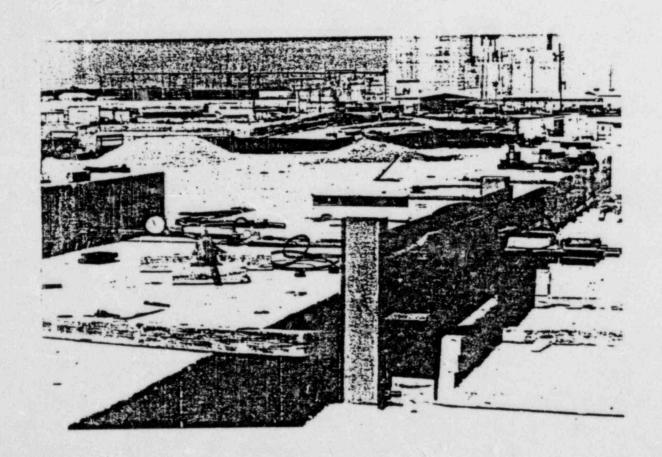
LOAD-DEFLECTION CURVES I-INCH TYPE EC-2W TENSION TEST



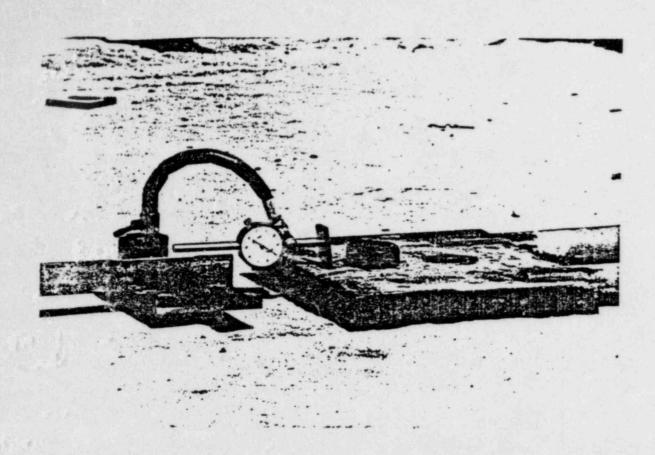
APPENDIX 4

PICTURES OF ACTUAL TEST APPARATUS

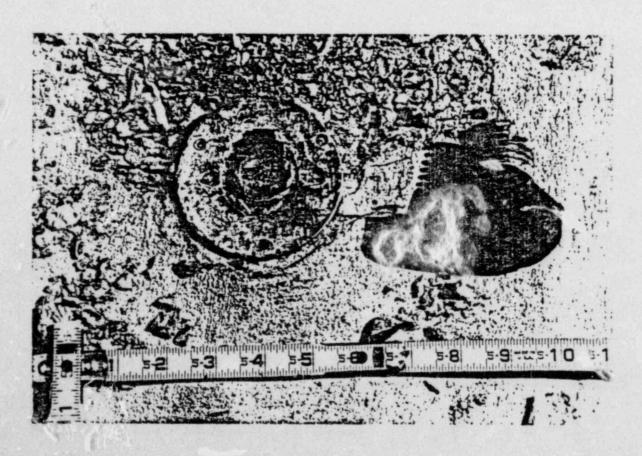
SHEAR TEST



TEST APPARATUS

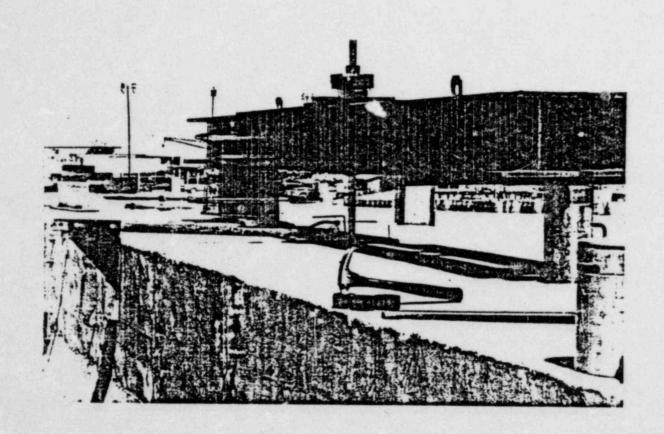


DIAL INDICATOR ARRANGEMENT

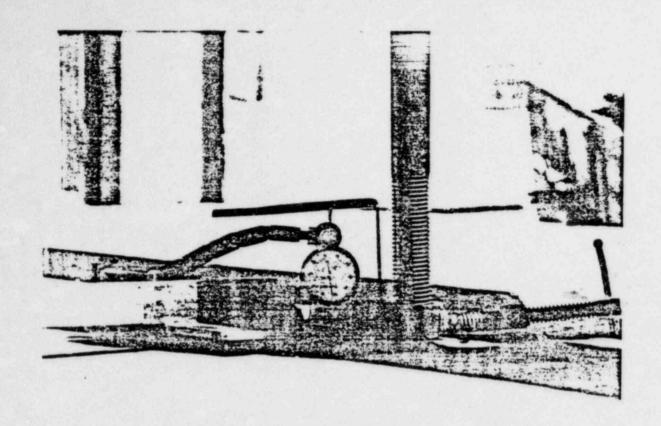


TYPILAL SHEAR FAILURE

TENSION TEST



TEST APPARATUS

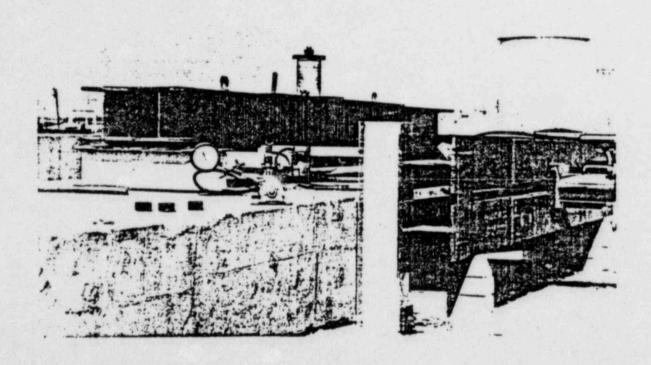


DIAL INDICATOR ARRANGENENT

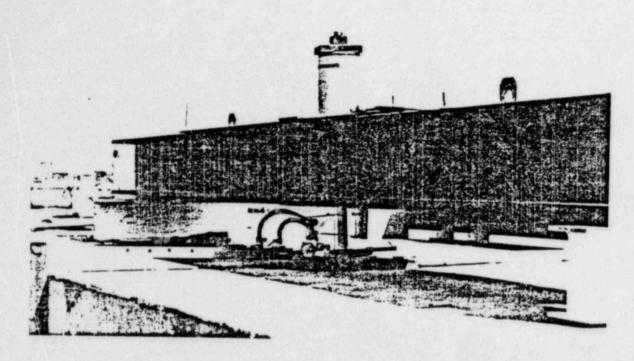


CONCRETE SHEAR CONE FAILURE

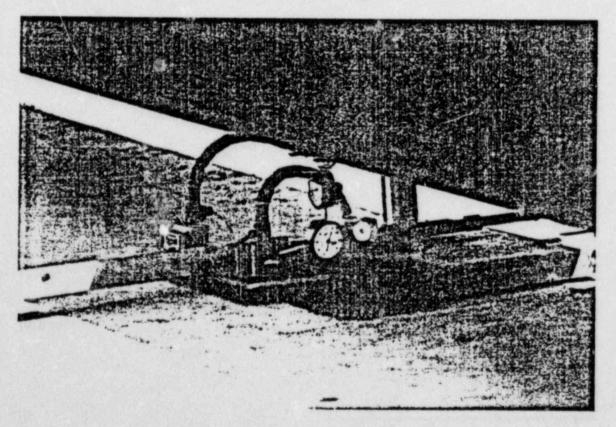
COMBINED SHEAR AND TENSION



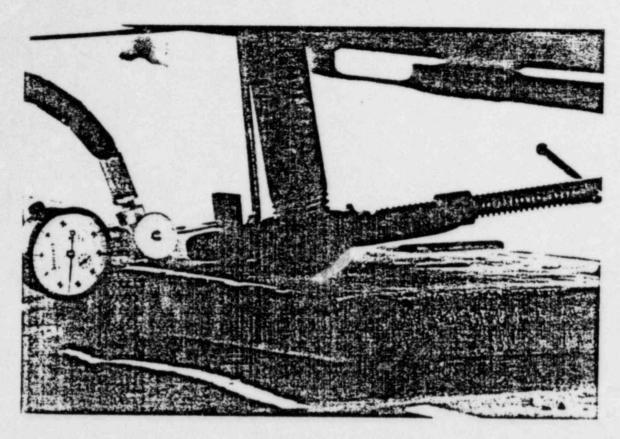
TEST APPARATUS



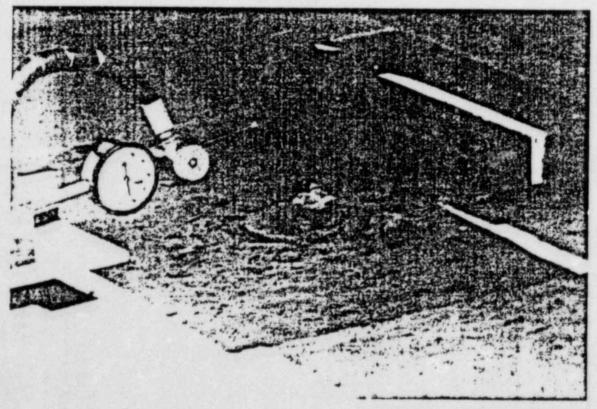
TEST AFPARATUS



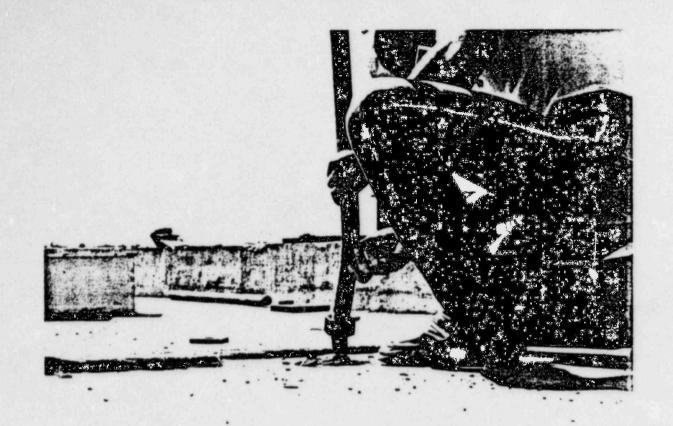
DIAL INDICATOR ARRANGEMET



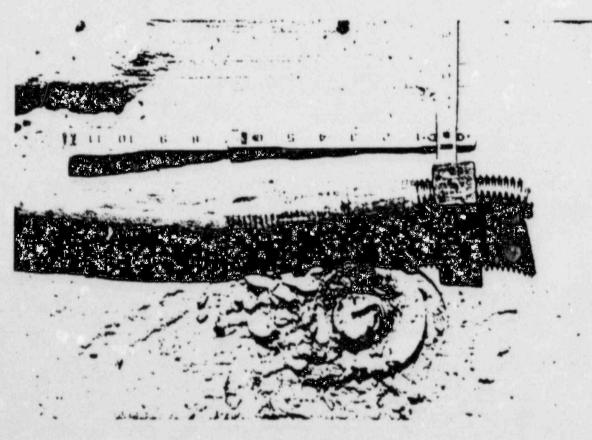
1/2-INCH SPECIMEN JUST PRIOR TO FAILURE



1/2-INCH SPECIMEN AT FAILURE

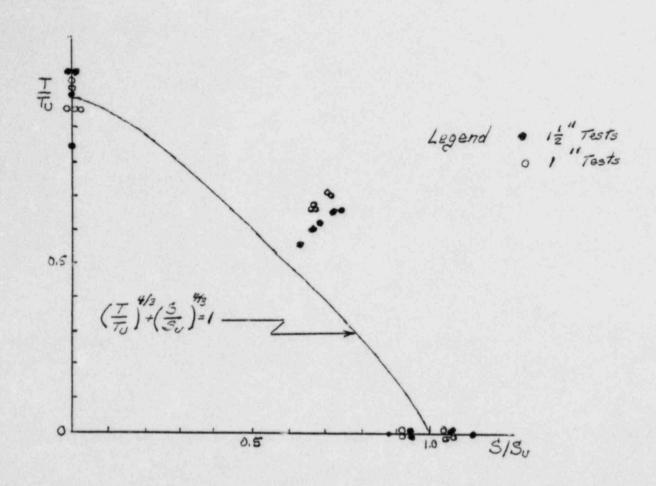


12-INCH FAILED SPECIMEN



TYPICAL FAILURE

ATTACHMENT C COMBINED SHEAR AND TENSION TEST SUMMARY



ATTACHMENT D

ORIGINAL DESIGN APPROACH AND COMPARISON WITH MEMORANDUM AND ORDER

When a Richmond insert assembly (tube steel, washers, bolt and insert) is subjected to a torsion in the tube steel, T, the additional tension P resulting in the bolt is computed (original design) as follows assuming that the bolt is originally tensioned to a value equal to Q Equilibrium equations (for symbols refer to Figure D-1)

The force Q can also be written as f_s A_b where f_s is the stress in the bolt and A_b the bolt effective cross sectional area. Similarly the force C which acts at the distance of d_3 from the neutral axis can be written as $3f_c$ $\frac{1}{4}$ where $\frac{f_c}{2}$ is the average compressive stress of the concrete (for a triangular distribution), b is the width of the washer and 3/2 d_3 is the distance from the neutral axis to the edge of the washer (d_3 is the distance between the neutral axis and the centroid of the triangle which is at 2/3 of its base). Thus

$$P = f_s A_b + \frac{3}{4} f_c b d_3$$
 (1)

$$\leq M = 0$$
 $T = cd_2 = c(\frac{b}{2} - \frac{d_3}{2}) = 3/4 f_c b d_3(\frac{b}{2} - \frac{d_3}{2})$ (2)

The third equation employed is strain compatibility between the concrete and the bolt (note that this assumes that the distribution of stresses in the concrete is uniform and equal to that shown at all locations across the washer plate

Where E_c and E_s are the concrete and steel (bolt) moduli of elasticity. This leads, using $n=E_s/E_c$ to:

$$\frac{n^{\frac{1}{5}c}}{3/2 d_3} = \frac{f_5}{b/2 - 3/2 d_3}$$
 (3)

If one then replaces $\frac{3}{2}$ d₃ (the distance from the neutral axis to the edge of the washer) with X, and substitutes (1) and (2) into (3), the following equation is obtained.

Equation (4) is a cubic equation in X, which when solved yields the value of X and hence the location of the neutral axis. Once that value is known, the solution for the additional tension in the bolt can be solved from equation (1) recalling that X=3/2 d₃.

For the particular instance in which the bolt is subject to no preset tension, but the tube steel is subject to torsion, i.e. Q=0, equation (4) reduces to

$$\chi^2 + \frac{2n}{b} A_b \chi - n A_b \tag{5}$$

which can be solved for X, and yields

$$X = A_b + (-1 \pm \sqrt{1 + \frac{b^2}{mA_b}})$$
 (6)

Only one of the roots of equation (6) is appropriate. The solutions for X were tabulated by NPSI in their design methods and the tables were employed to compute the resulting bolt tension. For instance for $E_s/E_c=8$ and a 4 x 4 tube steel (b=4) with a bolt having an effective area of 0.606 in (1-inch bolt) one would obtain

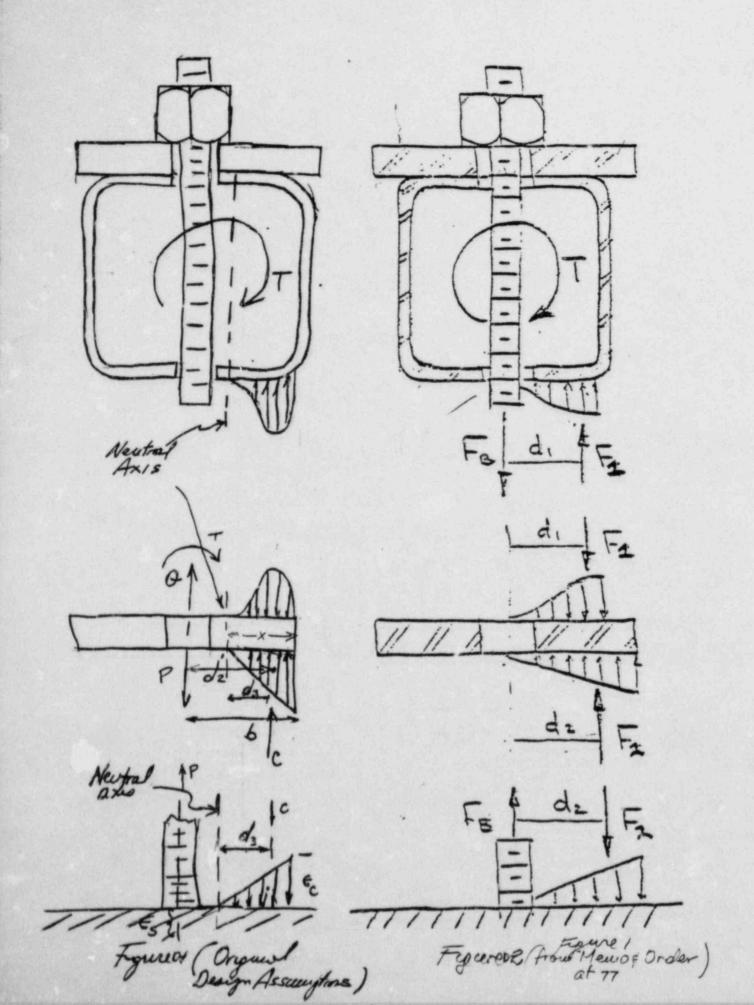
This means that the neutral axis is shifted from the bolt centerline 0.699 inches in the direction of the applied torsion.

Another interesting fact about equation (6) is that the location of the neutral axis is <u>independent</u> of the applied torsion. If there had been continuity between the bolt, the washer and the concrete (as for instance in an embedded plate with welding between the washer and the plate) the condition that the neutral axis is purely dependent on the moduli of elasticity of steel and concrete would probably be satisfied. In retrospect, after the Board's Order, it was this result that led us to suspect the validity of the strain compatibility equation and the development of the finite element model solution.

The difference brought about by the finite element analysis is best explained by the following: in the original design calculation, the computation of the tension in the bolt by

equation (1) is entirely equivalent to taking a lever arm from the center of the bolt to the centroid of the triangular compressive stress distribution in the concrete. This can be verified by noting that with the assumptions made in equation (1), (2) and (3), that centroid occurs at a distance $(\frac{b}{2} - \frac{x}{3}) = d_2$ From equation (2) and equation (1) we can write 3/4 for $bd_3 = f_3A_b + P$ and $T = (f_3A_b - P)(\frac{x}{2} - \frac{d_3}{2})$ But $x = 3/2d_3$, thus $T = (f_3A_b + P)d_3$

Hence for the case in which $Q = fs A_b = 0$ we have $T = Pd_2$ This is what Applicant had used. What the finite element analysis indicated is that the correct formula should be $T = Pd_4$, where d_4 is the distance from the bolt to the point of tangency between the tube steel and the washer.



ATTACHMENT E-2 RICHMOND INSERT - TUBE STEEL ASSEMBLY

FINITE ELEMENT ANALYSIS

A. Analysis of Richmond Insert Assembly

Ts 4 x 4 x 3/8 Tube with 1 1/2" Dia. Bolt - Radius = 2t

Eccentricity = 0"

INTRODUCTION

Ts 4 x 4 x 3/8 with 1 1/2" Dia. Bolt is used for the analysis because, except for a few 1/2" thick tubes Ts 4 x 4 x 3/8 represents the worst condition with respect to torsion.

A Richmond insert assembly was modeled with a $1\ 1/2$ " dia. Bolt at the center line of assembly as shown in Figure E-2. The purpose of this model is to study the behavior of the assembly and also concrete reactions for various loading conditions.

The analysis is performed using 'STARDYNE' computer program.

FINITE ELEMENT MODEL

A finite element model consisting of a Ts $4 \times 4 \times 3/8$ tube with two inserts is used. The spacing between the two inserts is taken to be 20" and the tube is modeled with an outer radius of 2t = 3/4").

Advantage is taken of the symmetric nature of the geometry and loading. Therefore, only half of the complete geometry is used. However, proper boundary conditions are enforced in the plane of the symmetry. The tube and the two 1" washer plates are modeled using either triangular or quadrilateral plate elements. The model is shown in Figure E-1, (a) through (f). The concrete reactions are obtained from the 'SPRING' subprogram of 'STARDYNE' which uses non-linear springs. The spring constant for concrete is calculated based on the theory of elastic half space. These ground springs are tied to the '3000' series nodes and are shown on E-1 (d). This drawing also shows the rigid beams that connect from the center line of the top washer plate to the surface of the tube steel given by '1000' series beams and from the center line of the top washer to the concrete surface by '2000' series beams. Rigid beams numbering B-1 to B-99 extend from the center line of top of tube and are shown on E-1 (c). The top, bottom and sides of the tube are modeled with triangular or quadrilateral plate elements and are shown on E-1 (a) and E-1 (b). The bolt is modeled by using beam elements. But in practice the bolt will behave

differently because of its very small span to depth ratio. This is discussed in Attachment E-3. The interface between the top of tube and top washer plate is modeled in such a way that only compression is transferred. If any rigid beam in this interface is found to carry tension, they are softened and removed so as not to transfer any tensile load. This is an itterative process and is used to obtain the final solution. The three loads (1) Pure torsion (2) Shear at center line of tube along 'Z' axis and (3) Shear ('Z' axis) and torsional moment are applied at the center of span (=20") shown as center line section on E-1 (c).

RESULTS AND DISCUSSION

The results of the three cases namely

(1) Pure torsion

(2) Shear at center line of tube, and

(3) Shear and torsional moment at center of tube

are detailed in Figure E-2 pages (a) through (e).

PURE TORSION (Load 1)

A torsional moment of 4000 in. lbs. is applied at center of 20" span through nodes 544, 555, 560 & 564 shown in center line Section on E-1 (c).

Two conditions are analyzed:

(a) There is a clearance between the bolt and tube
 (b) There is no clearance between the bolt and tube (i.e. bolt bearing against the tube).

The results are compared with case (c) which is the value obtained by using three design equations of Attachment D and are shown on Figure E-2 (a).

ANALYSIS OF THREE CASES

LOAD 1 (Bolt with clearance)

The applied torsional moment is resisted by a couple produced by compression in the concrete and tension in the bolt. The arm of the resisting couple being the distance between the center line of the bolt and the tangent point of the round corner. With a radius of 2 x thickness, arm = 2 - 2x.375 = 1.25".

The transfer of forces between the tube and top washer plate takes place along a line corresponding to the tangent point

of the interface. These forces are plotted in Section D-D, on E-2 (c). Except for the extreme two spikes the contact forces are relatively uniform.

The concrete reaction forces are shown in Section (1)-(1), (5)-(5) and (9)-(9) on E-2 (b). Maximum forces being at the edge and reducing toward the center.

LOAD 1
Case (B) (Bolt bearing against the tube - no clearance)

The applied torsional moment is resisted by the combination of a bolt tension/concrete compression couple and a moment in the bolt. The arm for the couple is same as in Case (A). The transfer of forces between the tube and top washer plate, top washer plate and concrete is similar to Case (A). This condition is an extreme case and provides an upper bound value for the moment in the bolt. Normally the bolt would not contact the tube steel because the lateral displacement of the tube steel at node 261 shown on E-1 (b) is only 0.0035" whereas there is a nominal all around gap of 1/16".

LOAD 1 Case (C)

The axial value shown is obtained by the use of three design equations. The value obtained from the finite element analysis (Case A) is 18.3% higher than the value for Case C.

Shear at Center of Tube (Load 2)

A shear load of 1000# is applied along 'Z' global axis at modes 546 and 561 shown in center line section on E-1 (c). Because of the applied shearing force, the clearance between the bolt and the tube is assumed to have closed.

Applied shear causes a turning moment which is resisted by the combination of the couple produced by compression of concrete and pull in bolt and by the moment in the bolt itself. These results are shown in Case (a) of E-2 (d).

A comparison with the current design method of analysis is shown in Case (b). In the current design method, an equivalent pull based on the three design equations and calculated from torsional moment of 1500 lb. in. caused by the lateral force would be used. The 1500 in-lb torsional moment is caused by the shear force of 500 lb acting 3 in above the concrete on each bolt.

Transfer of forces between the tube and top washer plate, top washer plate and concrete is similar in nature to Load (1) as shown in E-2 (b) & E-2 (c).

Shear and Torsional Moment at Center of Tube (Load 3)

A shear force of 1000# is applied at center of 20 in. span through node 566 shown in center line section on E-1 (c). The node 566 is 2" off the face of Ts 4 x 4 x 3/8 tube. This case is basically a combination of Load 1 & 2. The torsional moment caused by the lateral load is resisted by the combination of the couple and the moment in the bolt similar to Load 2. Transfer of forces between the tube and the top washer plate, the top washer plate and the concrete is similar in nature to Load 1. These results are shown in E-2 (e) Case (A).

A comparison with the current design method of analysis are shown on Case (B). In the current design method an equivalent pull based on three design equations and calculated from a torsional moment of 3500 lb. in is used.

B. Analysis of Richmond Insert Assembly Ts 4 x 4 x 3/8 Tube with 1 1/2" dia. Bolt Radius = 2t Eccentricity = 3/4"

The finite element model and its method of analysis is the same as in part (A) except that model is modified to move the bolt hole to an eccentricity of 3/4".

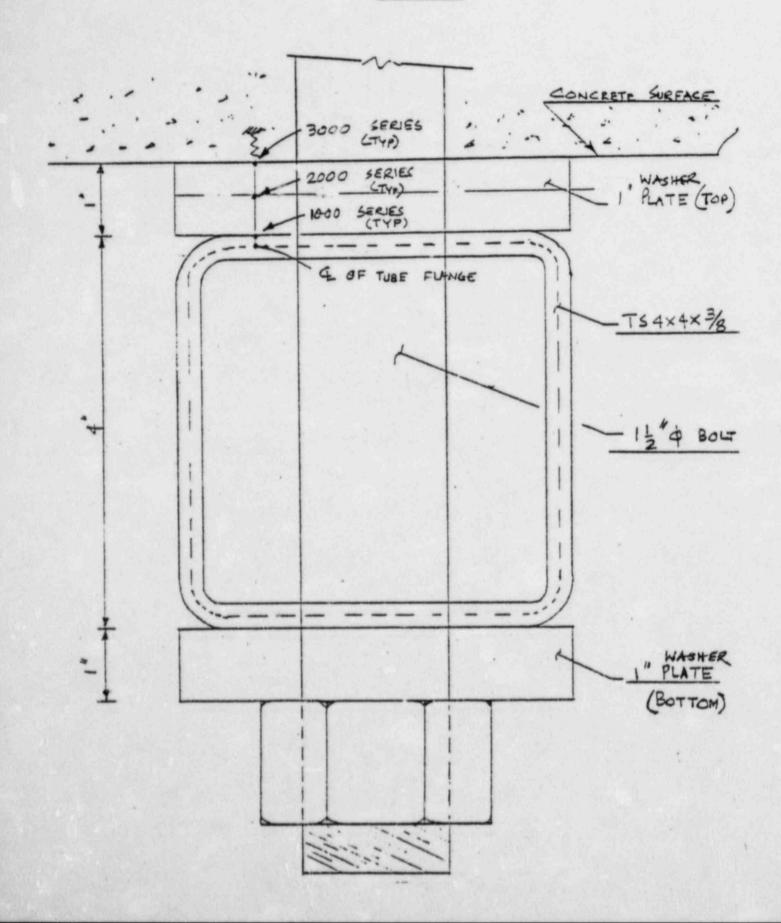
Load points and the three load cases are same as in Part (A). 3/4" eccentricity is used to understand the behavior of the assembly and to determine the limiting value of eccentricity.

Results and Discussions

For all three cases of loading, all the applied loads are resisted by the bolt itself. The resisting couple provided by the compression in concrete and tension in bolt, which is evident in non-eccentric condition has disappeared due to the very small lever arm. The applied torsional moment is transferred by shear couple produced by lateral forces due to rotation of tube against the bolt.

Ts $4 \times 4 \times 3/8$ with $1\frac{1}{2}$ " Dia. Bolt (e = 0")

Figure E-2



Ts 4 x 4 x 3/8 Tube with 1½" Dia. Bolt

e = 0"E - 2 (a)

		BOLT	OLT REACTION (12" P)		
LOAD 1 (PURE TORSION)	CASE	AXIAL (LB)	MOMENT (LB IN)	SHEAR (LB)	
20	@	1600	0	0	
MT = 4000 IN	&	986	767	0	
		1353	0	G	

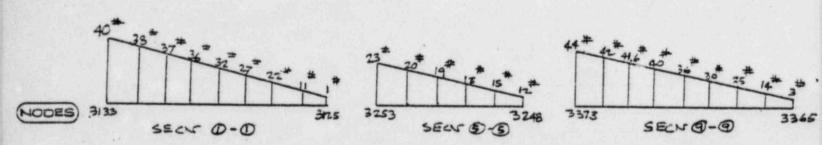
^{*} Ratio of a to c is 1.18. The 18% increase is less than what would have been obtained by ratioing the lever arm from bolt centerline to the tangent line to the old lever arm used (neutral axis to center of triangular distribution) which is 25%.

Ts $4 \times 4 \times 3/8$ TUBE WITH $1\frac{1}{2}$ " dia. BOLT e = 0"

E-2 (b)

CONCRETE REACTION (Case A) Load I

These values are obtained from the finite element analysis for ground spring nodes shown on E-I (d). Reactions for two boundary sections (I)-(I) and (9)-(9) and the center line section (5)-(5) are plotted to show the trend of compressive forces. The values shown are not to scale. These sections (I)-(I), (5)-(5) and (9)-(9) are shown in E-I (a) & E-I (e).



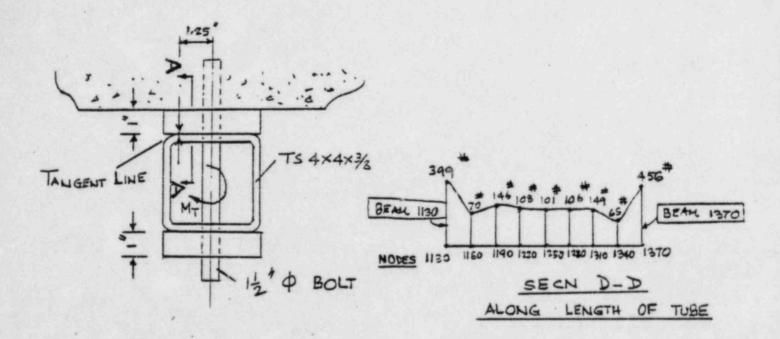
Only Two End Nodes Are Shown For All Sections

Due to three dimensional nature of the problem, the concrete reaction forces goes around the bolt.

Ts $4 \times 4 \times 3/8$ With $1\frac{1}{2}$ " dia. Bolt e = 0" E-2 (C)

Force Transfer Between Top of Tube and Top Washer Plate Case (A) Load I

These values are obtained from the beams connecting the tube to top washer plate interface. Only compressive forces are transferred through these beams. These beams are shown in E-I (d) and the nodes are shown in E-I (c). The values shown are not to scale. Beams 1160 to 1340 exists in Section D-D between beam 1130 & 1370.



The spikes shown at the ends are from the fact that concrete reactions are not uniform and are higher near the end section as shown in (1)-(1), (5)-(5) and (9)-(9) shown in E-2 (b). The finite element analysis shows that only the beams (1130 to 1370) along the tangent line carry the compressive forces.

Ts $4 \times 4 \times 3/8$ Tube with $1\frac{1}{2}$ " Dia. Bolt

 $\frac{e = 0"}{E - 2 (d)}$

LOAD 2	BOLT REACTION (12" 4)			
(SHEAR AT CENTER OF TUBE)		AXIAL	MOMENT *	SHEAR (LS)
20"	Case ②	507	866	500
1000	Core	1015		500

^{*} Moment in holt is set up by a shear couple with approximately 85 percent of the shear going to the upper tube steel face and 15 percent going to the lower tube steel face.

Ts $4 \times 4 \times 3/8$ with $\frac{1}{2}$ " Dia. Bolt e = 0"

E - 2 (e)

LOAD 3	BOLT REACTION (12"\$)			
(SHEAR & TORSIONAL MOMENT)	CASE	(La.)	(LB-IV)	SHEAR (LG)
20	@	1500	1620	500
1000	©	2368		500

FINITE ELEMENT ANALYSIS OF BOLT (11 DIA.) FOR TS 4 × 4 × 3/8 TUBE USING SOLID ELEMENTS

INTRODUCTION

A Richmond insert assembly has been analyzed using a finite element model whose analysis and results are provided in Attachment E. The purpose of the model was to determine the behavior of the assembly for various loading conditions. The I½" diameter bolt is modeled as a beam element. The finite element result for all three load cases in Attachment E,(1) Pure torsion but bolt leaning against the tube,(2) Shear at center of tube and (3) Shear and torsional moment at center of tube, show some moment being resisted by the bolt.

Because of small span to depth ratio the behavior of the bolt will differ from the condition where simple theory of flexure for a cantilever beam can be readily applied to determine bending stresses. In order to determine the magnitude of stresses caused by lateral loading of the bolt, a finite element analysis of the 1½" dia. bolt is performed using solid elements via STARDYNE program.

FINITE ELEMENT MODEL

The bolt length between the center of (tube) bottom flange and face of concrete is divided into seven slabs of varying thickness and shown in E-3 (a) through (i). The last slab near the concrete face is ‡" thick and shown on E-3 (h) & (i).

The base of the bolt is connected to the insert through springs with same spring constant used in the Attachment 'E' model ie Ts $4 \times 4 \times 3/8$ with $\frac{1}{2}$ " dia. bolt and zero eccentricity. A typical connection at base is shown on E-3 (i).

A 1000# lateral load is applied along global 'Z' (X_3) direction through nodes 24, 25 & 26 shown in E-3 b) to represent load from bottom flange.

RESULTS & DISCUSSIONS

Applied Moment = 1000×4.8125 = 4812.5 lb. in. Using simple bending theory

For 11 0 Bolt based on gross area

Area = 1.7671 in.²

Diameter = 1.5 inch

Section Modulus = $0.098175 \times (1.5)^3$ = 0.331 in.^3

Bending stress = $\frac{4812.5}{0.331}$ = 14539 p.s.i.

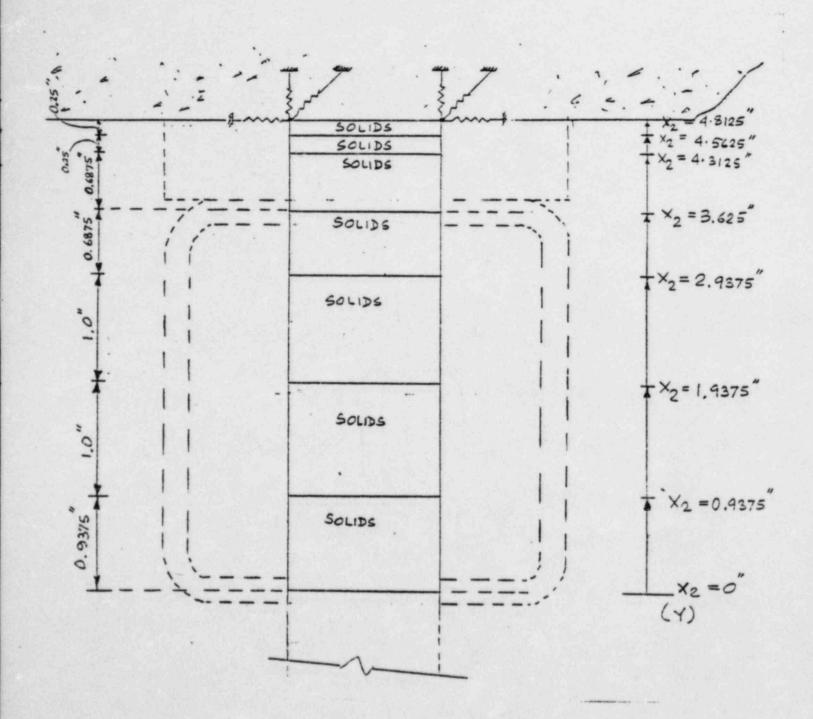
Based on finite element results the average stress across the furthest node (311) shown on E-3 (i) is about 10,836 p.s.i. This stress value is obtained by averaging the results of the elements (287), (297), (307) & (317).

Comparing the results it can be seen that stress obtained from finite element analysis is much less than that obtained from simple flexure theory. Hence it can be concluded that simple flexural behavior is not the case in this bolt and MC alone, without modification should not be used to

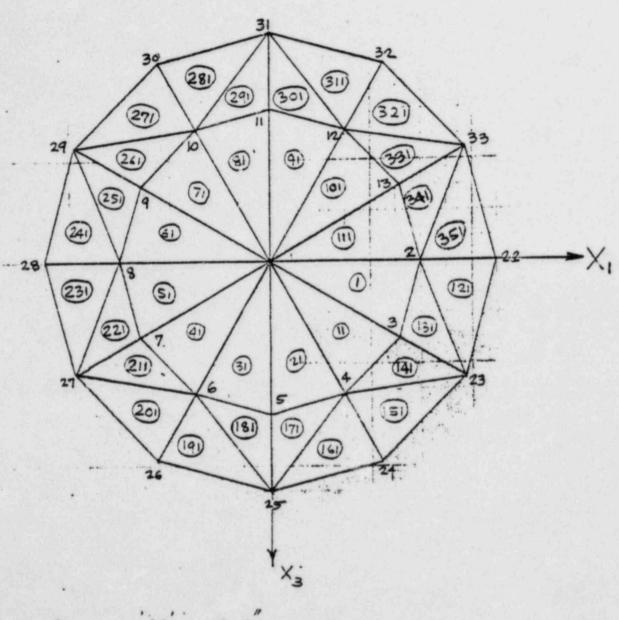
calculate bending stress. Actually the 14524 p.s.i. stress calculated would be higher if it was calculated on the basis of finite element model area which is 1.687 in. 2 and not 1.7671 in. 3 as used for comparison.

FINITE ELEMENT ANALYSIS FOR 13" DIA. BOLT

For Ts $4 \times 4 \times 3/8$ Tube Using Solid Elements E - 3 (a)

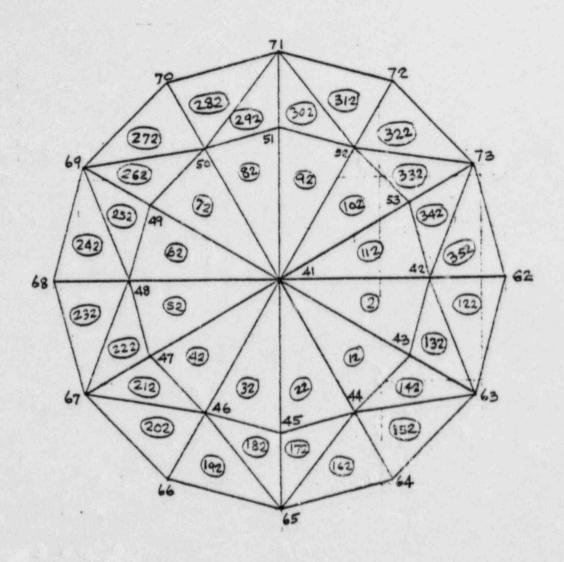


FINITE ELEMENT ANALYSIS OF 13" DIA. BOLT E - 3 (b)



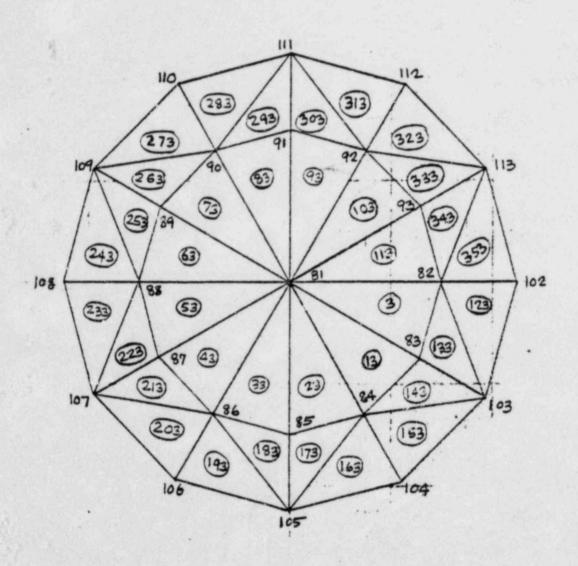
PLANE X2 = 0.0"

FINITE ELEMENT ANALYSIS OF 11 DIA. BOLT E - 3 (c)



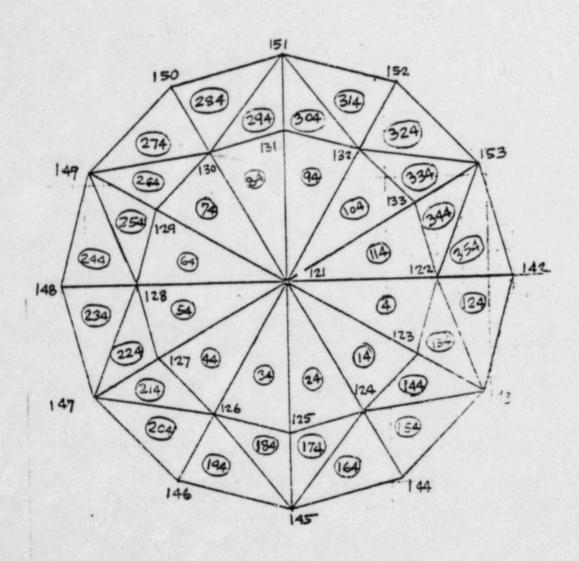
PLANE 'X'= 0.9375"

FINITE ELEMENT ANALYSIS OF 11" DIA. BOLT E - 3 (d)



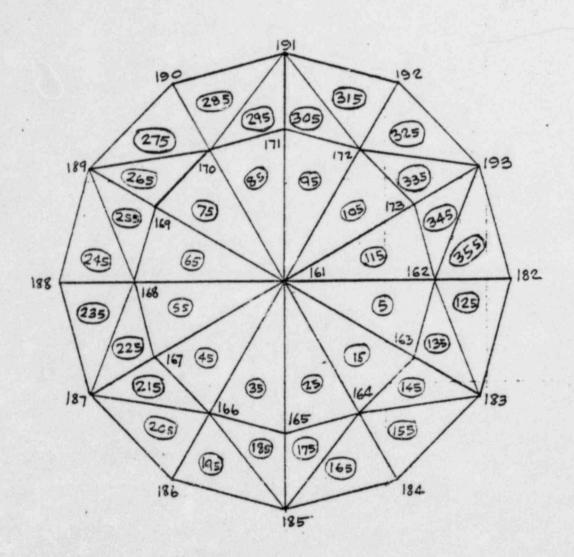
PLANE X2 = 1.9375"

FINITE ELEMENT ANALYSIS OF 11 DIA. BOLT E - 3 (e)



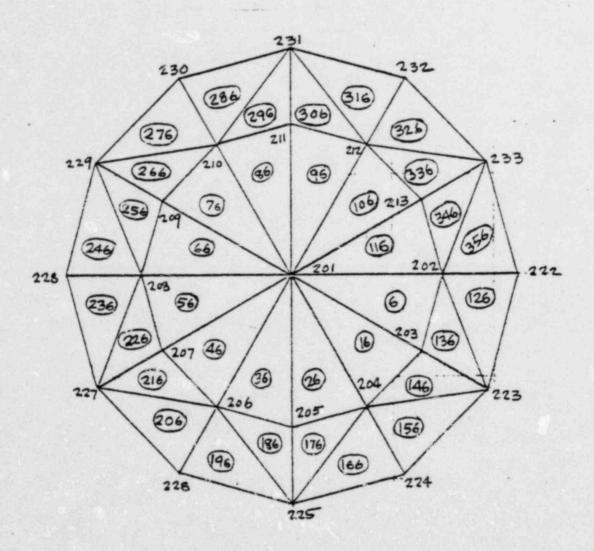
PLANE X2 = 2.9375

FINITE ELEMENT ANALYSIS OF 12" DIA. BOLT E - 3 (f)



PLANE 'X2 = 3.625"

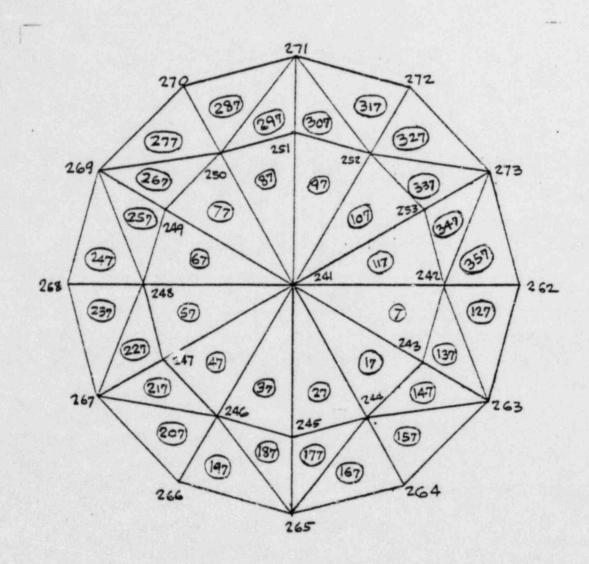
FINITE ELEMENT ANALYSIS OF 12" DIA. BOLT E - 3 (g)



PLANE 'X' = 4.3125"

FINITE ELEMENT ANALYSIS OF 13" DIA. BOLT

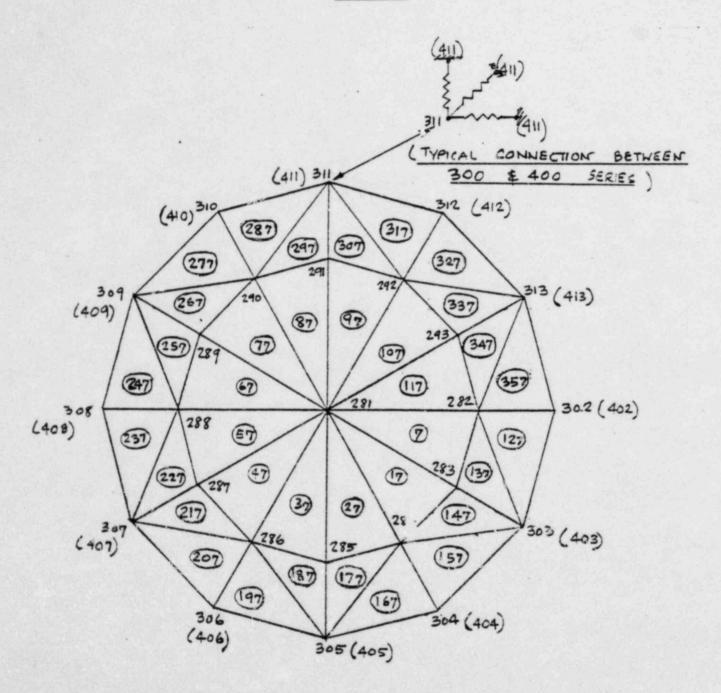
E - e(h)



PLANE X= 4.5625"

FINITE ELEMENT ANALYSIS OF 11" DIA. BOLT

E - 3 (i)



PLANE X2 = 4.8125"

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ATTACHMENT F

RICHMOND & TUBE STEEL
LOADING IN SHEAR
& TORSION

MAY 1984

1. Test Description

The following tests were performed on four Richmond Inserts/Tube Steel connections:

- o Shear load applied to 6"x6"x1/2" Tube Steel with bolt hole on TS centerline Test No. 1
- o Shear load applied to 4"x4"x3/8" Tube Steel with bolt hole offset 3/4" from the TS centerline Test No. 2
- o Torsional load applied to 4"x4"x3/8" Tube Steel with bolt hole on TS centerline Test No. 3
- Torsional load applied to 4"x4"x3/8" Tube Steel with bolt hole offset 3/4" from TS certerline Test No. 4

 Figure 1 shows photographs of the test set up and the final configurations of the assemblies after the test. Attachments

 F-1, F-2, F-3, and F-4 provide results for the four tests.

2. Summary of Results

Table F-1 presents a comparison of the test results with the following 4 Insert Design Methods. Columns A through D refer to each of the methods listed below:

- Method A. This method assumes the torsion is resisted by a couple whose moment arm is 2/3 the half width of the washer plate.
- Method B. This method assumes the torsion is resisted by a couple whose moment arm is that predicted by strain compatibility as described earlier in Attachment D.

- Method C. This method assumes the torsion is resisted by a couple whose moment arm is the distance from the bolt centerline to the point of tangency between the tube steel and the washer plate.
- Method D. This method assumes the torsion is resisted partially as described in Method C above, and partially by bending of the bolt. This is the method utilized in generating the interaction ratios shown in Table F of this Affidavit.

Table F-1 also contains the Design Loads based on the insert and bolt capacities for the four methods and a factor of safety for these Design Loads based on the test results. The table also provides the tube steel deflection for the various loads.

3. Conclusions From Test Results

The test results indicated that little or no deformation of the tube steel occurs at loads corresponding to the design loads. The tests also indicate that the initial design methods have a factor of safety in excess of 3. They further indicate that the point of tangency methods has a factor of safety in excess of 4 when bolt bending is neglected and a factor of safety in excess of 12 when bolt bending stress is considered by calculating it using MC/I where M is the bending moment, C the bolt diameter and I the bolt moment of inertia. The test results indicate that the failure mechanism is by shear type deformation for the 6x6 TS

shear test (Case 1) and by bolt bending for the 4x4 TS with 3/4" eccentricity (Case 2 and 4) and Case 3 (4x4 TS with 0 eccentricity loaded in torsion).

Cases 2, 3 and 4 were designed to be analogous to the finite element analysis discussed previously in Attachment E so that they could be used to validate the following conclusions reached from the analysis.

The finite element analysis predicts that for the 4x4 TS with high eccentricity loaded either in shear or torsion, the bolt bending governs the design. The test verified that this is the failure mechanism, however, the failure load predicted by the test is considerably higher than that predicted by the finite element analysis. The analysis predicts that failure of an elastic-perfectly plastic round section loaded in bending is 2 1/4 times the load which produces a bending stress of .75 Fy (Fy is the yield strength). This load is defined as the Design Load. The test results indicate that the actual load for the bolt is 12.5 to 12.8, or about 5 times higher than the Design Load. This discrepancy is due in part to the conservatism involved in using MC/I to calculate the bending stress in the bolt. This conservatism is determined by comparison with the results of the bolt finite element analysis. It is due in part to the assumption of elastic perfectly plastic behavior of the bolt material, which in reality strain hardens, and it is also due in part to the assumption that all the torsional moment is carried by the bolt in bending. Although this is what the finite element

analysis predicts, some of the torsional moment is taken by a bolt tension/concrete compression couple, particularly at the higher loadings where the deformation of the tube steel provides a compressive area that establishes the couple. Since the finite element analysis is purely elastic, once some local yielding occurs, the analysis would not predict the redistribution of the torsional moment to the tension/compression couple that would result in higher load capacities. The discrepancy is also due in part to the fact that the finite element analysis, in predicting the bolt moment due to shear, does not account for redistribution of the shear between the upper and lower tube steel as deformation occurs. In addition, the discrepancy is due in part to the fact that friction is not included in the analysis. In summation, all of the above factors show why the test results verify that the calculation of the design capacity using a method based on the finite element analysis is very conservative.

The other two test cases also demonstrate that the calculation of the design load based on finite element analysis is also very conservative regardless of tube steel size and eccentricity for the same reasons as stated above.

When the test results are compared to either of the initial design methods (A or B), the test shows that the design load capacities of these methods have reasonable factors of safety and, therefore, there is no safety concern with the initial design methods.

In addition, comparison of the test results with method C which neglects bending of the bolt, shows that there is no concern if bending of the bolt is ignored.

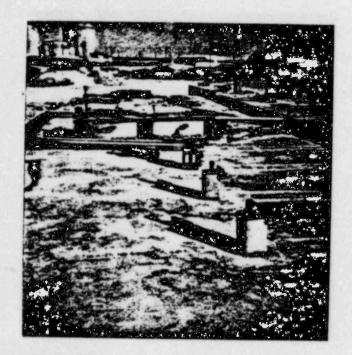
In summary, the test results demonstrate that the original design methods used for the design of the connections were adequate and that the design method based on the finite element analysis is very conservative.

TABLE F-1

	CASE	TEST ULTIMATE CAPACITY	METHOD A 2/3×1/2 WIDTH	METHOD B STRAIN COMPATIBILITY	METHOD C POINT OF TANGENCY W/O BOLT BENDING	METHOD D POINT OF TANGENCY W/ BOLT BENDING	
	6x6x1/2	46.37	11.00	12.04	11.00	2.45	Max. Design Capacity
0 Offset Shear		4.2	3.8	4.2	18.9	FS = Test Ultimate Capacity + Design Capacity	
			•09	-10	•09	-01	Tube steel deflection at design capacity based on test curve.
	4x4x3/8	23.85	7.14	NA*	4.53	1.91	Max. Design Capacity
	3/4 Offset Shear		3.3	_	5.2	12.5	FS = Test Ultimate Capacity * Design Capacity
			-07		•02	.01	Tube Steel deflection at design capacity based on test curve
	4×4×3/8	25.17	5.124	5.62	4.828	1.38	Max. Design Capacity
	0 Offset Torsion		4.9	4.4	5.2	18.2	FS = Test Ultimate Capacity + Design Capacity
			.07	.07	•07	< .01	Tube steel deflection at design capacity based on test curve
	4x4x3/8	10.6	3.28	NA*	1.99	.824	Max. Design Capacity
	3/4 Offset Torsion		3.23	_	5.32	12.8	FS = Test Ultimate Capacity * Design Capacity
			•02		.07	<.01	Tube steel deflection at design capacity based on test curve

Loads are in kips, Deflections are in inches

^{*}The strain compatibility method was used only for eccentricities < 3/8"



TEST ASSEMBLIES PRIOR TO TESTING

5/11/84



TORSION TEST ON 4X4 TS
WITH HOLE ON TS &
TEST #3



SHEAR TEST ON 6x6 TS WITH HOLE ON TS & TEST#1



TEST ASSEMBLIES
AFTER TESTING

5/11/84



SHEAR TEST ON YX4 TS WITH HOLE 3/4" OFF-SET TEST#2

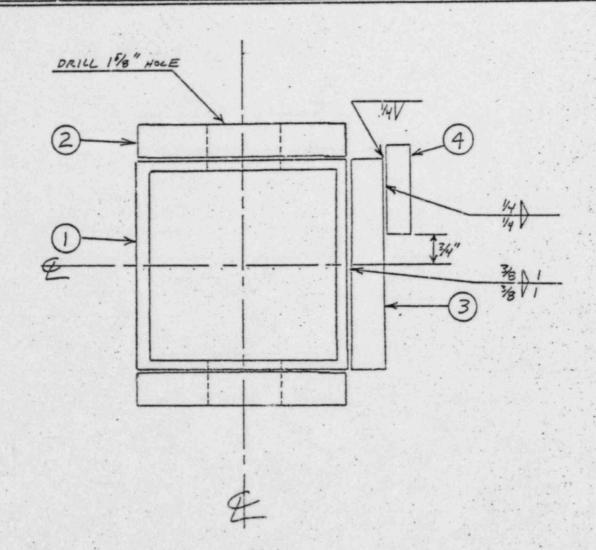


TORSION TEST ON 4X4 TS WITH HOLZ OFF-SET 34"
TEST #4

BROWN & ROOT, INC.

SUEEL	nu.	
	COLUMN TO SERVICE	

		0	ENGINEERING	DIVISION			
CLIENT	TUSI					_ 103 NO	116
	RICHMOND	INSERT	SHEAR	TEST			y Carron
BASED ON _					DRAWING NO.		100 TH
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- 1) TS 6x6"x1/2" 6" LONG
- 2 A-36 PLATE 1"X6" 6" LONG (2 REQ'D)
- 3) A-36 PLATE 1" X5"- 5" LONG
- 4) A-36 PLATE 3/4"X2" 4" LONG
- (5) A-36 THREADED ROD 1/2" \$ 16" LONG

ENGINEERING DIVISION ATTACHMENT F-1

P 2/3 SHEET NO. _ 1 OF

CLIENT TUSI _____ JOB NO. _____ SUBJECT SHEAR TEST ON 6x6 TS WITH & HOLE ON TS & DRAWING NO. BASED ON ____ COMPUTER TR CHK'O. BY APP'O. BY DATE MAY 11 DNUT GUAGE AREA FORCE | ATS* REMARKS PRESSURE 13.25 IN2 6,625 # .049" . 046 " 500 PSI 13,250 # 13,250 # . /35" -108" 1,000 PSI . 236" .179" 13.25 NY 19, 875# 1,500 PSE 13.25 NY 26, 500 # . 395" 2,000 PSI . 286" 13,25 NL 33,125# . 619" . 430" 2,500 PSI 13.25 N. 39, 750 * . 698" 1.041" 3,000 PSI 1.865" 13.25,N2 43,725# 1.254" 3,300 PSI EXCEED TRAVEL 13,25 IN 46, 375# 1. 829" 3,500 PSI ON LOWER NUT COULD NOT ATTAIN 3,600 PS "X" PSI DTS - TAKEN & COATS DNUT - TAKEN P & OF LOWER WUT

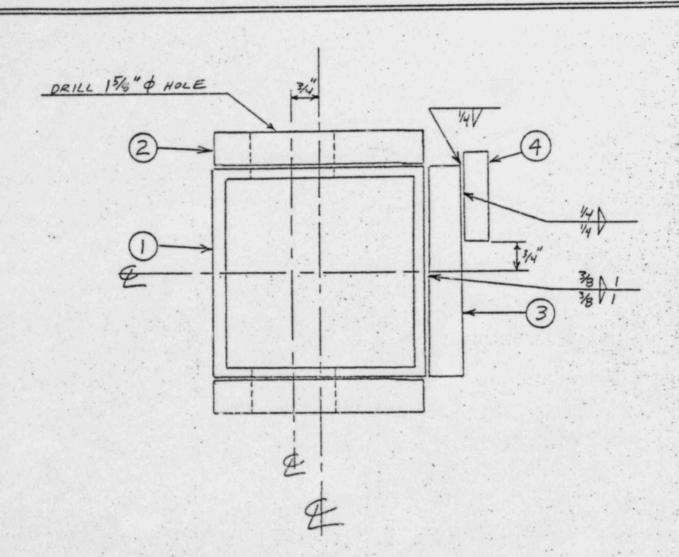
ATTACHMENT F-1 P. 3/3 BROWN & ROOT, INC. SHEET NO. 2 OF_ CLIENT TUS I JOB NO. ___ SUBJECT SHEAR TEST ON 6X6 TS WITH & HOLE ON TS DRAWING NO.__ BASED ON ____ COMPUTER TR 40,000 50,000 1 TS 40,000 30,000 20,000 10,000 2.0 .5 1.0 1.5 DEFLECTION (IN.)



BROWN & ROOT, INC.

ENGINEERING DIVISION

SHEET NO.___



- 1) TS 4"x4" x 3/8" 6" LONG
- (2) A-36 PLATE I"X4"-4" LONG (2 REQ'D)
- (3) A-36 PLATE 1"X3"-5" LONG
- 4) A-36 PLATE 3/4" × 1"4" 4" LONG
- (5) A-36 THREADED ROD 12"6 14" 12NI

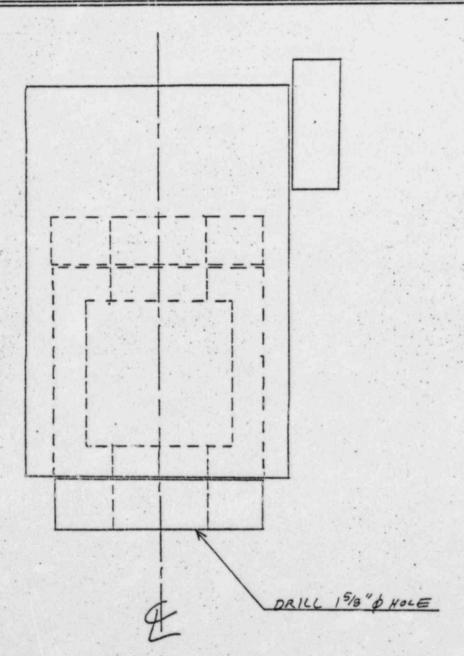
ATTACHMENT F-2



BROWN & ROOT, INC.

SHEET NO. 2

CLIENT	TUSE	the control of the				108 NO
SUBJECT	RICHMOND	INSERT	TORSION	TEST		have brisidly
BASED ON _					DRAWING NO.	والمحاجب والا
COMPUTER _	TR	CHK'0. BY	APP'D.	ву	DATE	MAYI



TEST # 2 BROWN & ROOT, INC. ATTACHMENT F-2
ENGINEERING DIVISION P.3/4 SHEET NO. ______ P. 3/4 SHEET NO. ____ OF CLIENT_TUSI ____ JOB NO. ____ SUBJECT SHEAR TEST ON 4 X4 TS WITH 3/4" OFF-SET OF HOLE & DRAWING NO. COMPUTER TR CHK'D. BY APP'D. BY DATE MAY 11 PRESSURE AREA FORCE | ATS DNUT 1 REMARKS 13.25 IN2 6,625# .068" . 095" 500 PST 13,25/14 13,250# .228" .130" 1,000 PSE 13.25 INL 19, 875# . 439" 1.116" 1,500 PSI EXCEEDED TRAV .605" 13.25 my 22,525# ON DIAL INDICAT 1,700 PSI ON LOWER NUT 1,800 PSE 13.25 IN2 23,850# .918" * COULD NOT ATTA 1,400 PSI "X" PSZ * 13.25 14 "X" 1.210" * ATS - TAKEN @ COFTS * AMT - TAKEN @ & OF COWER WLIT



BROWN & ROOT, INC.

ATTACHHENT F-2
P 4/4
SHEET NO. 2 OF _

BLECT SHEAR TE	ST ON YX4 TS	WITH 3/4" OFF	-SET OF HO	LEE
SED ON			DRAWING NO	
MPUTER TR	CHK'D. BY	APP'D. BY	DATE	97 11
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DEFLECTION (IN.)

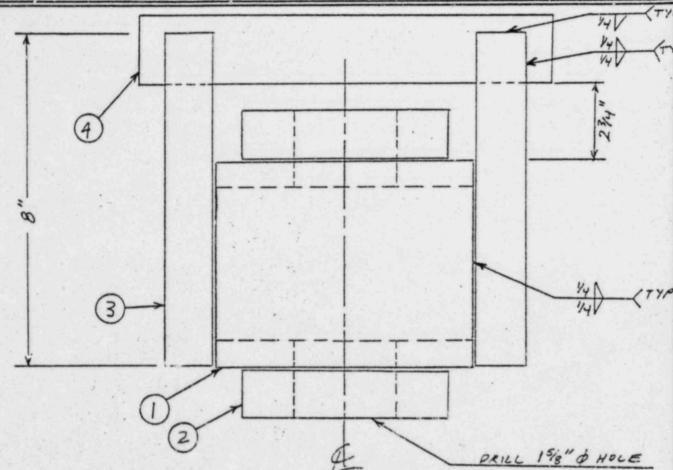
TEST#3



BROWN & ROOT, INC.

ENGINEERING DIVISION

CLIENT	TUSI				_ JOB NO	
SUB/ECT	RICHMOND	INSERT	TORSION	TEST		
BASED ON				DRAWING NO.		
COMPUTER	IR CH	K'D. BY	APP'O. BY	DATE	MAY	1
COMPUTER _	IR CH	K'D. BY	APP'D. BY	DATE	MAY	_



- 1) TS 4"x4" x 3/8" 41/2" LONG
- 2 A-36 PLATE I"X4"-4" LONL (2 REQ'D)
- 3 A-36 PLATE 1"X5"-8" LONG (2 REQ'O)
- A-36 PLATE 1" x 21/2" 7/2" LONG
- (5) 4-36 THE THE TOTAL TOTAL 1/4 " # 111" 1 11

TE	57 #3						A	TTAC	HMEN	17 F-3
CLIENT TUS.	₹	(6)			& ROO			7.	2/3	NOOF
SUBJECT TORSION		ON	4×4	TS	WITH	HOLE	4	ON		
BASED ON								ING NO.		

BASED ON	-0				WING NO.
COMPUTER _ A	CHK.	D. 8Y	APP'0. B		DATE MAY 11 1
PRESSURE	AREA	FORCE	A 75	DNUT	REMARKS
200 PSI	13.251,12	2,650#	.009"	.019"	
500 PSI	13,25 INZ	6,625#	. 118"	.156"	
700 PSI	13.25 TN2	9,275#	.200 "	.322"	
1,000 PSE	13.25 NZ	13,250 *	.29/"	. 516"	
1,200 PSE	13,25 THE	15,900=	.380"	.73/"	
1,500 PSI	13.25 NL	19,875#	. 591"	1.268"	
1,700 PSI	13.25 NL	22,525#	.766"	1.688"	
1,800 PSI	13,25 INL	23,350#	. 932"	_	EXCEEDED TRAVE
1,900 PSI	13.25 INC	25,175#	1.134"	_	
"X" P3_T*	13.25 /	" x "	1.351"		* COULD NOT ATTAIL 2,000 PSI
*075	- TAKE	104	OFTS		
* D NUT		P & 0	FLOWER	NUT	



BROWN & ROOT, INC. SHEET NO. 2 OF _ TUSI SUBJECT TORSION TEST ON 4X4TS WITH HOLE & ON TS DRAWING NO.____ BASED ON __ COMPUTER TR 60,000 50,000 40,000 30,000 20,000 10,000 .5 2.0 1.5 1.0

BROWN & ROOT, INC.

P.1/4 SHEET NO. 1 OF

CLIENT	TUSI				_ JOB NO
SUBJECTBASED ON		OND INSERT	TORSION TES	DRAWING NO.	
		CHK'O. 8Y	APP'O BY	DATE	MAY 1
					141/ (7)
1				-	14)
	/	1			1
	4				284"
					<u> </u>
200					
Ĭ					
					44 (77)
	3	→			14) (77)
1					
		(2)			
			4	PRILL 1	88" \$ HOLE

- 1 TS 4"x4" x 3/8" 41/2" LONG
- (2) A-36 PLATE I"X4"-4" LONG (2 REQ'D)
- 3 A-36 PLATE 1"X5"-8" LONG (2 REQ'O)
- 4 A-36 PLATE I"X 2'2"- 7'2" LONG
- 5 A-36 THREADED ROD 1/2" \$ 14" LONG

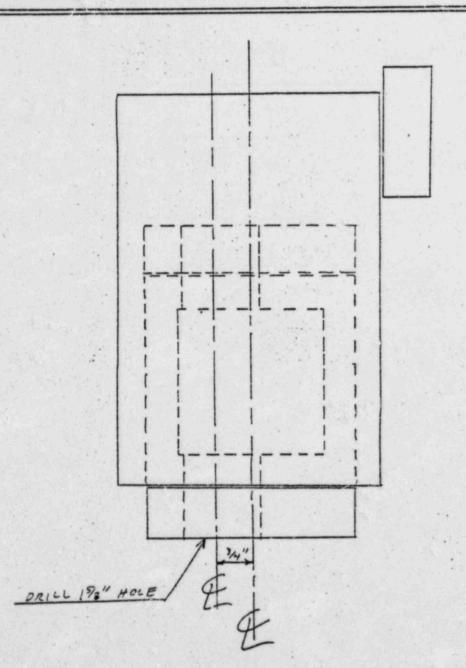
TEST #4

ATTACHHENT F-4



BROWN & ROOT, INC. P. 2/4 ENGINEERING DIVISION

CLIENT TUST _ JOB NO. __ SUBJECT_ RICHMOND INSERT TORSION TEST DRAWING NO.___ BASED ON ___ COMPUTER TR CHK'D. BY APP'D. BY DATE MAY!





ATTACHHENT F-4

	- 6	(3)	ENGINEERING DI	VISION	P.3/4 SHEET NO OF
CLIENT TU	5.2-		T.C	3/1" 055	SET OF HOLE
	ON TEST	0 7 7 7 7	13 WITH		ANTING NO.
COMPUTER	-RCHX ·	D. 8Y	APP'O. B		DATE MAY 11
PRESSURE	AREA	FORCE	Δ 7's	D. T.	REMARKS
200 PSI	13.25 IN	2,650#	.043"	.087"	
400 PSI	13,25,102	5,300#	.104"	.210"	
SOO PSE	13.25 TH2	6,625#	.254"	.530"	
600 PSE	13.25 TAR	7,950	. 469"	1.019"	
700 PSE	13.25 INZ	9,275#	.608"	1.353"	
800 PSE	13.25 WZ	10,600#	.806"	1.829"	
"X" P5 = *	13.25 N	"×"	1.205"		* COULD NOT ATTAIN 900 PSI
* ATS -	TAKEN	@ ¢ 0	= TS		
* A NUT-				NUT	
1					
			 		

((13)

BROWN & ROOT, INC.

.

P. 44 SHEET NO. 2 OF

CLIENT TUS I SUBJECT TORSION TEST ON YXY TS WITH 3/4" OFF-SET OF HOLE & DRAWING NO._ COMPUTER TR 30,000 25,000 20,000 15,000 10,000 5,000 2.0 1.5 1.0

DEFLECTION (IN.)

RAY PETER DEUBLER PROJECT MANAGER - NPS INDUSTRIES

EDUCATION

B.S., Mechanical Engineering, Cornell University, 1969. M.E., Mechanical Engineering, Cornell University, 1970.

· EXPERIENCE

Mr. Deubler has 14 years of experience in the area of Mechanical Engineering. Mr. Deubler is currently Project Manager for NSPI for the Comanche Peak Project. As such he is responsible for all NPSI Design and Fabrication activities for this project.

Previously Mr. Deubler was Director of Engineering at NPSI and as such he supervised the engineering, development and qualification of standard pipe support components, field service activities, and the design and fabrication of supports including their conformance to ASME Section III.

Earlier at NPSI, Mr. Deubler supervised the design and fabrication of piping supports for various projects in both the nuclear and fossil industries. His responsibilities included the overall supervision and management of the design, fabrication, and detailed engineering work on all phases of the design, fabrication and quality assurance aspects of component supports.

Mr. Deubler was an Instrumentation Engineer at Gibbs and Hill. Principal work was performed in control valves, instrumentation, control systems, and components for power plants. Other work included the selection, specification, and procurement of components as well as the designing of instrumentation and control loops for fluid systems.

At the American Electric Power Service Corporation, Mr. Deubler was Mechanical Engineer in charge of specifying, selecting, and purchasing piping equipment for major power plant projects including valves, piping, supports, and miscellaneous piping systems components. He also designed plant fluid systems.

Other experience includes design work in the areas of plumbing and HVAC for Buchart and Horn.

PROFESSIONAL

Professional Engineer, New York.

Member of ASME and AWS.

Member of working group on component supports of Subcommittee III of the ASME Boiler and Pressure Vessel Committee.

Member of Committee 8C3 - Pipe Hangers and Supports of Manufacturers Standardization Society of the Valve and Fitting Industry (MSS).

TABLE 1 PART A

SUPPORT MARK NO.	TUBE STEEL SIZE	ECCENTRICITY	BOLT SIZE	INSERT INTERACTION	BOLT INTERACTION	
AF-1-006-010-S33R	8 x 4 x 3/8"	1 3/4"	1"	.51	2.45	FE
CC-1-008-013-S33K	8 x 4 x 1/2"	7/16"	1"	1.53	4.32	FE
CC-1-197-005-C52R	4 x 4 x 3/8"	0	1 1/2"	.21	.924	
CC-1-197-014-C42R		0	1 1/2"	.196	.3786	
CC-1-197-019-C52R	4 x 4 x 3/8"	1/8	1 1/2"	.32	.53	
CC-1-197-020-C52R	4 x 4 x 3/8"	0	1 1/2"	.143	.309	
CC-1-197-034-C52R	4 x 4 x 3/8"	1/16	1 1/2"	.19	.89	
CC-1-204-003-C52R	4 x 4 x 3/8"	0	1 1/2"	.239	.673	
C-1-205-016-C53R		0	1 1/2"	.059	.779	
CC-1-206-001-C53R		0	1 1/2"	.259	.779	
CC-1-207-014-C53R		0	1 1/2"	.029	.171	
C-1-207-021-C53R		0	1 1/2"	.102	.031	
C-1-212-001-C53R		5/8	1 1/2"	.12	.70	
C-1-215-032-C53R		0	1 1/2"	.07	.30	
CC-1-215-033-C53R		0	1 1/2"	.03	.17	
C-1-217-003-C53K		0	1 1/2"	.21	.43	
C-1-217-012-C53S		3/8"	1 1/2"	.10	.995	
C-1-218-009-C53K		0	1 1/2"	.15	.33	
C-1-218-010-C53K		0	1 1/2"	.011	.07	
C-1-218-012-C53K		1/8"	1 1/2"	.75	.14	
C-1-218-013-C53K	The state of the s	0	1 1/2"	.223	1.74	
C-1-218-014-C53K		0	1 1/2"	.04	.38	
C-1-226-004-C53R			1 1/2"	.31	.61	
C-1-226-005-C53R		0 0 0	1 1/2"	.13	.42	
C-1-227-003-C53R		0	1 1/2"	.26	.56	
C-1-231-002-C53R		0	1 1/2"	.015	.06	
C-1-233-001-C53R		3/4"	1 1/2"	.084	.7	
C-1-233-004-C53R	Name of the Party		1 1/2"	.073	.27	
C-1-234-016-C53R		0	1 1/2"	.18	.44	
C-1-237-001-C53R			1 1/2"	.06	.41	
C-1-237-004-C53R			-		-	
C-1-239-005-C53R	4 x 4 x 3/8"	1/4	1 1/2"	.03	.30	
C-1-239-008-C53R			_		27	

FE - Requires further evaluation.

SUPPORT MARK NO.	TUBE STEEL SIZE	EXCENTRICITY	BOLT SIZE	INSERT INTERACTION	BOLT INTERACTION
CC-1-242-002-C53R	4 x 4 x 3/8"	3/8"	1 1/2"	.03	.32
CC-1-242-003-C53R	4 x 4 x 3/8"	1/2"	1 1/2	.02	.32
CC-1-245-010-C53R	4 x 4 x 3/8"	0	1 1/2	.03	.38
CC-1-245-018-C53R	4 x 4 x 3/8"	0	1 1/2	.009	.16
CC-1-249-003-C53R	4 x 4 x 3/8"	0	1 1/2	.26	.35
CC-1-249-700-C53R	3 x 6 x 5/16"	0	1 1/2"	.12	.48
CC-1-255-007-C53R	6 x 6 x 3/8"	3/8	1 1/2"	.37	1.17
CC-1-271-008-C53R	4 x 4 x 3/8"	0	1 1/2"	.26	.81
CC-1-272-008-C53K	4 x 4 x 3/8"	0	1 1/2"	.024	.176
CC-2-040-401-A33K	4 x 4 x 3/8"	1/16	1"	.27	.52
CC-2-040-405-E33R	6 x 4 x 1/2"	C	1"	.54	1.72
CC-2-048-402-A33R	6 x 6 x 1/2"	0	1"	.52	.70
CC-2-048-403-A33R	6 x 6 x 1/2"	0	1"	.08	.52
CC-2-048-408-A33K	6 x 6 x 3/8"	0	1"	.29	1.17
C-2-105-406-E23P	6 x 6 x 3/8"	0	1"	.06	.43
CC-2-107-403-E23S	4 x 6 x 3/8"	0	1"	.23	.52
CS-1-001-003-C42K	4 x 4 x 3/8"	0	1 1/2"	.008	.058
CS-1-001-011-C42R	4 x 4 x 3/8"	0	1 1/2"	.16	.44
CS-1-001-012-C42R	4 x 4 x 3/8"	0	1 1/2"	.36	.61
CS-1-001-024-C42K	4 x 4 x 3/8"	0	1 1/2"	.15	.45
CS-1-001-027-C42K	4 x 4 x 3/8"	0	1 1/2"	.15	.386
CS-1-001-035-C42R	4 x 4 x 3/8"	0	1 1/2"	.24	.67
CS-1-012-003-C42R	4 x 4 x 3/8	0	1 1/2"	.11	.39
CS-1-077-004-C42R	4 x 4 x 3/8"	0	1 1/2"	.0315	.1498
CS-1-077-005-C42R	4 x 4 x 3/8"	5/16"	1 1/2"	.03	.30
CS-1-077-006-C42R	4 x 4 x 1/4"	0	1 1/2"	.05	.31
CS-1-078-003-C42R	4 x 4 x 3/8"	0	1 1/2"	.24	.92
CS-1-078-018-C42K	6 x 6 x 3/8"	0	1 1/2"	.048	.43
CS-7 -079-006-C42R	6 x 4 x 3/8"	0	1 1/2"	.062	.6
CS-1-079-007-C42R	4 x 4 x 3/8"	0	1 1/2"	.48	.65
CS-1-079-020-C42R	6 x 6 x 3/8"	C	1 1/2"	.13	.51
CS-1-079-037-C42K	4 x 4 x 3/8"	1/4	1 1/2"	.09	.64
CS-2-033-408-A42R	4 x 4 x 3/8"	0	1 1/2"	.073	.037
CS-2-085-402-A42S	4 x 4 x 3/8"	5/8	1"	.06	.31
CT-1-018-005-S22R	4 x 4 x 3/8"	0	1"	.17	.8
CT-1-038-003-C52R	4 x 4 x 3/8"	0	1 1/2"	.037	.15
CT-1-038-402-C52R	4 x 4 x 3/8"	0	1 1/2"	.037	.15

FE - Requires further evaluation.

TABLE 1 PART A

SUPPORT MARK NO.	TUBE STEEL SIZE	ECCENTRICITY	BOLT SIZE	INSERT INTERACTION	BOLT INTERACTION	
CT-1-038-415-C62R	6 x 6 x 3/8"	13/16"	1"	.30	.15	
CT-1-038-430-C62K	4 x 4 x 3/8"	1/4	1 1/2"	.25	.59	
CT-1-038-431-C52R	4 x 4 x 3/8"	0	1 1/2"	.053	.18	
CT-1-039-008-C42R	4 x 4 x 3/8"	0	1 1/2"	.07	.21	
CI-1-039-020-C42R		3/4	1 1/2"	.112	.81	
CT-1-039-402-C42S	5 x 5 x 3/8"	13/16	1 1/2"	.02	.17	
CT-1-039-405-C42S	4 x 4 x 3/8"	0	1 1/2"	.2	.144	
CT-1-039-407-C42R	4 x 4 x 3/8"	0	1 1/2"	.19	.31	
CT-1-039-413-C42A		0	1"	1.4	3.03	FE
CT-1-039-415-C42R	4 x 4 x 3/8"	0	1 1/2"	.24	.93	
CT-1-039-424-C42R	4 x 4 x 3/8"	0	1"	.28	.69	
CT-1-039-432-C42K		1/8	1 1/2"	.09	.086	
CT-1-039-433-C42K	4 x 4 x 1/4"	0	1 1/2"	.358	.506	
CT-1-039-434-C42R		0	1 1/2"	.209	.477	
CT-1-039-435-C42K	CT-1-039-402-C42S		-			
CT-1-039-436-C42R	4 x 4 x 3/8"	5/16"	1 1/2"	.07	.58	
CT-1-039-445-C42R	4 x 4 x 3/8"	0	1"	.21	.82	
CT-1-039-447-C42R	4 x 4 x 3/8"	0	1"	.351	.99	
CT-1-051-406-C72K	4 x 4 x 3/8"	1/2"	1 1/2"	.024	.285	
CT-1-053-408-C62R	4 x 4 x 3/8"	0	1 1/2"	2.13	3.88	FE
CT-1-053-418-C62R	6 x 6 x 3/8"	0	1 1/2"	1.48	4.12	FE
CT-1-054-401-C42R	4 x 4 x 3/8"	1/4	1"	.17	1.26	
CT-1-054-404-C42R	4 x 4 x 3/8"	1/2	1 1/2"	.083	.616	
CT-1-054-406-C42R	6 x 6 x 3/8"	1 13/32	1 1/2"	.06	.21	
CT-1-054-409-C42K	4 x 4 x 3/8"	0	1 1/2"	.26	.364	
CT-1-054-413-C42R	€ x 6 x 3/8"	1/2	1 1/2"	.09	.86	
CT-1-054-420-C42R	6 x 6 x 3/8"	1"	1 1/2"	.17	1.49	
T-1-054-424-C42R	4 x 4 x 3/8"	0	1"	.54	.61	
T-1-054-429-C42R	4 x 4 x 3/8"	5/8	1 1/2"	.0975	.51	
CT-1-054-430-C42R	4 x 4 x 3/8"	3/8 "	1"	2.78	8.41	FE
CT-1-054-431-C42A	6 x 6 x 3/8"	1/2"	1"	.55	3.39	FE
CT-1-054-438-C42R	4 x 4 x 3/8"	0	1"	.23	.63	
CT-1-054-442-C42R	4 x 4 x 3/8"	0	1 1/2"	.03	.219	
CT-1-117-403-C62R	4 x 4 x 3/8"	1/8"	1 1/2"	.11	.80	

FE - Requires further evaluation.

SUPPORT MARK NO.	TUBE STEEL SIZE	ECCENTRICITY	BOLT SIZE	INSERT INTERACTION	BOLT INTERACTION	
CT-1-117-404-062R	6 x 6 x 3/8"	1/8"	1 1/2"	.05	.22	
CT-1-117-405-C62K	The second state of the second	0	1 1/2"	.077	.394	
CT-1-117-410-C62K		1/2"	1 1/2"	.07	.52	
CT-1-124-412-C72K	4 x 4 x 3/8"	0	1 1/2"	.026	.21	
FW-1-097-018-C62R	6 x 6 x 3/8"	0	1"	1.39	7.93	FE
MS-1-002-004-C72K	6 x 6 x 1/2"	0	1 1/2"	.34	.44	
MS-1-002-005-C72K	6 x 6 x 1/2"	3/4"	1 1/2"	2.22	6.36	FE
MS-1-002-006-C72K	8 x 8 x 1/2"	0	1 1/2"	.47	.38	
MS-1-002-013-C72K	8 x 8 x 1/2"	0	1 1/2"	1.22	1.38	FE
MS-1-073-007-C52K	4 x 4 x 3/8"	0	1 1/2"	.145	.434	
MS-1-074-001- 352K	4 x 4 x 3/8"	1/8"	1 1/2"	.16	.43	
MS-1-074-002-C52S	4 x 4 x 3/8"	0	1 1/2"	.196	.25	
MS-1-074-003-C52K	MS-1-074-002-C525	3 -	-	-		
MS-1-074-010-C52K	4 x 4 x 3/8"	0	1 1/2"	.072	.33	
MS-1-074-012-C52K	4 x 4 x 1/2"	0	1 1/2"	.28	.52	
MS-1-150-002-C52S		3/16"	1 1/2"	.15	.5	1
MS-1-150-004-C52S	4 x 4 x 3/8"	1/2"	1 1/2"	.19	1.48	The same
MS-1-150-025-C52K	4 x 4 x 3/8"	0	1 1/2"	.095	.354	
MS-1-150-029-C52K	4 x 4 x 1/2"	3/16"	1 1/2"	.05	.4	
MS-1-150-044-C52R	6 x 6 x 3/8"	0	1 1/2"	.16	.68	DAME:
MS-1-150-045-C52K	4 x 4 x 1/2"	1/16"	1"	.187	1.56	
MG-1-150-058-C52K	4 x 4 x 3/8"	0	1 1/2"	.18	.53	
4S-1-150-059-C52K	MS-1-151-043-C52H	(-	-	-		
4S-1-150-064-C52K	MS-1-150-024-C521		1 1/2"	.65	.275	
4S-1-151-002-C52R	6 x 6 x 1/2"	1/8"	1 1/2"	.37	1.11	
15-1-151-005-C52R	4 x 4 x 3/8"	5/16"	1 1/2"	.27	1.15	
S-1-151-008-C52R	MS-1-150-010-C528	0	1"	.34	.39	
1S-1-151-018-C52R	4 x 4 x 3/8"	0	1 1/2"	.23	.67	
/S-1-151-019-C52R	4 x 4 x 3/8"	0	1 1/2"	.34	.95	
4S-1-151-038-C52R	5 x 5 x 1/2"	0	1 1/2"	.18	.74	
15-1-151-043-C52K	4 x 4 x 3/8"	1/2"	1 1/2"	.23	.66	
1S-1-345-005-C52K	4 x 4 x 3/8"	3/8"	1 1/2"	.07	.71	
S-1-416-005-S33R	6 x 6 x 1/2"	11/16"	1"	.64	3.42	FE
C-1-008-002-C41s	4 x 4 x 1/2"	3/8"	1 1/2"	.24	1.2	
C-1-018-020-C71R	6 x 6 x 1/2"	0	1 1/2"	.03	.31	
C-1-018-021-C71R	4 x 4 x 1/2"	0	1 1/2"	.17	.45	
C-1-075-044-C51K	6 x 6 x 1/2"	3/16"	1 1/2"	.22	1.09	
C-1-075-052-C61R	6 x 6 x 3/8"	0	1 1/2"	.6	1.66	

FE - Requires further evaluation.

TABLE 1 PART A

SUPPORT MARK NO.	TUBE STEEL SIZE	ECCENTRICITY	BOLT SIZE	INSERT INTERACTION	BOLT INTERACTION	
RC-1-087-004-C81K	6 x 6 x 3/8"	0	1 1/2"	.386	1.63	
RC-1-088-006-C81K	RC-1-087-001-C81	S 0	1 1/2"	.17	.61	
RC-1-162-004-C81K	6 x 4 x 1/2"	0	1 1/2"	.21	.67	
RC-1-164-001-C81K	6 x 6 x 3/8"	0	1 1/2"	.1426	.412	
RH-1-005-007-C42R	4 x 4 x 3/8"	0	1 1/2"	.023	.14	
RH-1-005-013-C42R	6 x 6 x 3/8"	1-3/8"	1 1/2"	.07	.94	
RH-1-006-010-C42K	6 x 4 x 3/8"	0	1 1/2"	.37	.77	
SI-1-051-012-C42K	6 x 4 x 3/8"	1/2"	1 1/2"	.11	.37	
SI-1-087-009-C42R	6 x 6 x 3/8"	1/8"	1 1/2"	.89	1.72	
SI-1-095-017-C42R	6 x 6 x 3/8"	0	1 1/2"	.46	1.22	
SI-1-102-007-C41R	6 x 6 x 3/8"	0	1 1/2"	.38	1.24	
SI-1-103-008-C42K	6 x 6 x 3/8"	0	1 1/2"	1.54	4.63	FE
SI-2-178-714-A32R	4 x 4 x 1/4"	0	1"	.26	.61	

FE - Requires further evaluation.

TABLE 1
PART B

SUPPORT MARK NO.	REV.	TENSION (KIPS)	SHEAR (KIPS)	MOMENT (KIP IN)
AF-1-006-010-S33R	2	1.398	.694	
CC-1-008-013-S33K	5	8.633	.46	1.021
CC-1-197-005-C52R	3	.113	.889	11.911
CC-1-197-014-C42R	3	6.0	.444	4.0
CC-1-197-019-C52R	2	2.0	.246	4.0
CC-1-197-020-C52R	3	2.0	.246	4.0
CC-1-197-034-C52R	3	1.0	1.0	4.0
CC-1-204-003-C52R	5	2.0	1.0	8.0
CC-1-205-016-C53R	4	2.0	2.0	2.0
CC-1-206-001-C53R	5	2.0	2.0	2.0
CC-1-207-014-C53R	4	-	.243	2.189
CC-1-207-021-C53R	4	1.0	1.0	4.0
CC-1-212-001-C53R	3	2.0	.247	2.0
CC-1-215-032-C53R	5	-	.256	4.0
CC-1-215-033-C53R		.25	0	2.5
CC-1-217-003-C53K	2	1.0	.5	4.0
C-1-217-012-C53R	1	-	.319	5.583
CC-1-218-009-C53K	2	1.0	.5	4.0
CC-1-218-010-C53K	4	-	.144	.335
CC-1-218-012-C53K	2	6.0	.02	2.0
CC-1-218-013-C53K	2	-	1.0	7.98
CC-1-218-014-C53K	2	1.0	1.0	4.0
CC-1-226-004-C53R	2	1.0	.5	8.0
CC-1-226-005-C53R	2 2 2 3 3	-	.487	5.511
CC-1-227-003-C53R	3	6.0	.17	4.0
CC-1-231-002-C53R	3	.235	-	.883
CC-1-233-001-C53R	2	.651	.337	2.321
CC-1-233-004-C53R	6	.4	-	4.0
CC-1-234-016-C53R	4	-	.879	5.264
CC-1-237-001-C53R	CC-1-235-001-C53R	2.917	.765	2.16
CC-1-237-004-C53R	CC-1-233-001-C53R	-	-	_
CC-1-239-005-C53R	3	.22		1.87
CC-1-239-008-C53R	CC-1-233-001-C53R	-		-

TABLE 1 PART B

SUPPORT MARK NO.	REV.	TENSION (KIPS)	SHEAR (KIPS)	MCMENT (KIP IN)
CC-1-242-002-C53R	3	.273	.273	.956
CC-1-242-003-C53R	3	0	.338	1.01
CC-1-245-010-C53R	2	1.0	1.0	4.0
CC-1-245-018-C53R	2 2	.204	.373	.714
CC-1-249-003-C53R	1	6.0	-	4.0
C-1-249-700-C53R	1	1.408	.817	4.496
CC-1-255-007-C53R	2	.433	1.59	0.0
CC-1-271-008-C53R	1	1.0	1.0	10.0
CC-1-272-008-C53K	2		.15	1.882
C-2-040-401-A33K	2	2.0	.073	1.751
CC-2-040-405-E33R	1	.266	1.3	5.65
CC-2-048-402-A33R	2	3.0	-	5.0
C-2-048-403-A33R	2	-	.493	0.0
CC-2-048-408-A33K	5	.04	.678	4.066
CC-2-105-406-E23P	4		.395	0.0
C-2-107-403-E23S	2	1.0	-	5.0
CS-1-001-003-C42K	6	.076	.127	.67
CS-1-001-011-C42R	6	.50	1.0	2.0
CS-1-001-012-C42R	6	6.0	1.0	6.0
CS-1-001-024-C42K	3	1.0	1.0	2.0
CS-1-001-027-C42K	3	2.0	1.0	4.0
CS-1-001-035-C42R	5 2 3	2.0	1.005	8.0
CS-1-012-003-C42R	2	.356	-	5.696
CS-1-077-004-C42R	3	.626	-	2.2
CS-1-077-005-C42R	3	.474	-	1.6
CS-1-077-006-C42R	3	.287	.114	3.957
CS-1-078-003-C42R	4	-	1.591	11.0
CS-1-078-018-C42K	3	-	1.028	0.0
CS-1-079-006-C42R	4	.5	.05	3.0
CS-1-079-007-C42R	6	2.0	1.005	8.0
CS-1-079-020-C42R	4	1.0	.50	4.5
CS-1-079-037-C42K	3	.975	0.0	3.862
CS-2-033-408-A42R	4	.98	.086	.667
CS-2-085-402-A42S		1.0	.895	6.0
CT-1-018-005-S22R	3 2	.186	0	3.314
CT-1-038-003-C52R	4	.50	.50	-
CT-1-038-402-C52R	4	.5	.50	_

TABLE 1 PART B

SUPPORT MARK NO.	REV.	TENSION (KIPS)	SHEAR (KIPS)	MOMENT (KIP IN)
CT-1-038-415-C62R	5	-	.38	2.115
CT-1-038-430-062K	5 3	6.0	.035	2.0
CT-1-038-431-C62R	3	.8	.4	2.0
CT-1-039-008-C42R	4	1.0	-	3.0
CT-1-039-020-C42R	2	1.394	.25	4.52
CT-1-039-402-C42S	4	.37	0	.674
CT-1-039-405-C42S	4	2.0	-	2.0
CT-1-039-407-C42R	4	4.0	-	4.0
CT-1-039-413-C42A	4	1.441	.55	1.0
CT-1-039-415-C42R	4	2.0	2.0	4.0
CT-1-039-424-C42R	3	.144	.425	2.676
CT-1-039-432-C42K	3	3.0	.20	2.0
CT-1-039-433-C42K	5	6.0	.180	6.0
CT-1-039-434-C42R	3	2.0	.48	6.0
CT-1-039-435-C42K	CT-1-039-402-C42S	-	-	-
CT-1-039-436-C42R	1	.515	.44	1.76
CT-1-039-445-C42R	3	.102	.72	2.458
CT-1-039-447-C42R	3	-	1.0	1.5
CT-1-051-406-C72K	4	.328	.031	1.312
CT-1-053-408-C62R	3	1.82	2.479	9.249
CT-1-053-418-C62R	3	3.0	1.72	46.068
CT-1-054-401-C42R	3	-	.304	1.357
CT-1-054-404-C42R	2	.50	-	2.0
CT-1-054-406-C42R	4	.383	0	1.186
CT-1-054-409-C42K	3	6.0	.187	4.0
CT-1-054-413-C42R	1	.1	.152	4.15
CT-1-054-420-C42R	3	1.5	1.025	3.0
CT-1-054-424-C42R	2	.523	-	4.7
CT-1-054-429-C42R	2	3.323		.786
CT-1-054-430-C42R	1	6.581	.274	9.889
CT-1-054-431-C42A	3	.227	.407	3.608
CT-1-054-438-C42R	2	-	.5	2.0
CT-1-054-442-C42R	2	-	.5	1.0
CT-1-117-403-C62R	3	.370	1.482	4.627

TABLE 1 PART B

SUPPORT MARK NO.	REV.	TENSION (KIPS)	SHEAR (KIPS)	MOMENT (KIP IN)
CT-1-117-4-4-C62R	2	.429	0	2.014
CT-1-117-405-C62K	5	.225	.04	5.43
CT-1-117-410-C62K	2	2.0	-	2.0
CT-1-124-412-C72K	4	-	.687	2.06
FW-1-097-018-C62R	3	1.46	1.95	36.941
MS-1-002-004-C72K	9	12.16	1.0	2.3
MS-1-002-005-C72K	7	19.37	2.37	9.80
MS-1-002-006-C72K	7	4.069	.455	2.048
MS-1-002-013-C72K	9	23.975	4.301	3.295
MS-1-074-001-C52K	4	2.128	.757	-
MS-1-074-002-C52S	5	2.0	-	4.0
MS-1-074-003-C52K	MS-1-074-002-052S	-	-	-
MS-1-074-010-C52K	2	.031	.372	4.253
MS-1-074-012-C52K	3	1.0	1.0	6.0
MS-1-150-002-C52S	2	.477	.09	6.55
MS-1-150-004-C52S	3	.246	.839	8.461
MS-1-150-025-C52K	5	.5	-	4.0
MS-1-150-029-C52K	4	.8	.2	2.2
MS-1-150-044-C52K	5	1.0	1.0	4.0
MS-1-150-045-C52K	4	-	.274	1.436
MS-1-150-058-C52K	3	2.0	1.0	5.0
MS-1-150-059-C52K	MS-1-151-043-C52K	-	-	
MS-1-150-064-C52K	4	6.0	.059	2.0
MS-1-151-002-C52R	5	6.0	.605	5.0
MS-1-151-005-C52R	4	3.0	1.3	5.62
MS-1-151-008-C52R	6	.186	.318	4.364
MS-1-151-018-C52R	6	2.0	1.0	8.0
MS-1-151-019-C52R	4	1.0	.2	13.0
MS-1-151-038-C52R	4	-	.71	7.174
MS-1-151-043-C52K	2	.014	1.64	6.614
MS-1-345-005-C52K	4	-	.27	3.309
MS-1-416-005-S33R	3	.812	-	5.027
RC-1-008-002-C41S	3	1.939	2.169	5.899
RC-1-018-020-C71R	6	1.497	.742	-
RC-1-018-021-C71R	4	1.373	1.308	6.0
RC-1-075-044-C51K	4	2.25	.334	7.821
RC-1-075-052-C61R	3	.998	2.291	10.212

TABLE 1 PART B

SUPPORT MARK NO.	REV.	TENSION (KIPS)	SHEAR (KIPS)	MOMENT (KIP IN)
RC-1-087-004-C81K	8	.095	1.5	15.0
RC-1-088-006-C81K	4	.327	.68	7.808
PC-1-162-004-C81K	5		2.36	10.0
RC-1-164-001-C81K	6	1.83	.082	5.49
RH-1-005-007-C42R	2		.18	1.27
RH-1-005-013-C42R	2	2.08	.05	5.253
RH-1-006-010-C42K	3	1.961	1.124	6.11
SI-1-051-012-C42K	3	1.497	1.655	.749
SI-1-087-009-C42R	3	2.67	1.55	9.84
SI-1-095-017-C42R	7	2.855	3.678	16.687
SI-1-102-007-C41R	7	-	.798	13.59
SI-1-103-008-C42K	5	3.933	10.63	2.306
SI-2-178-714-A32R	2	.664	.227	2.512

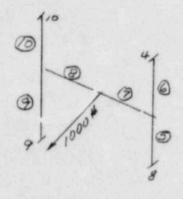
TABLE 2

TUBE STEEL & RICHMOND INSERTS COMPARISON OF RESULTS OBTAINED WITH STRUDL WITH AND WITHOUT RELEASING M

A. GENERIC STUDY

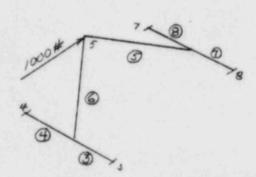
Problem I

Problem 2



All tube steel is 4"x4"x1/4"

I All moment constrained 2 All moments released



Member results

Member	Max. Stressi	Max. Stress2	Member	Max. Stressl	Max. Stress2
5	448.6	448.6	7	2729	2902
6	448.6	448.6	8	2729	2902
7	640.9	897.2	3	1453	1477
8	649.9	897.6	4	1453	1477
9	448.6	448.6	5	540	497
10	448.6	448.6	6	579	384
	Max. Increase	1/2 40%		Max. Increa	se 1/2 6\$
b. <u>D</u>	eflections at Pt. 5				
	.000902 .001184			.00569	•00607
	Max. Increase	1/2 = 31%		Max. Increa	se 1/2 = 7%
c. <u>T</u>	ension in Each Insert				
	838# 250#			1113#	500#
	Max. decrease	1/2 = 340\$		Max. ducrea	se 1/2 = 220¶

B. ACTUAL SUPPORTS

The following tube steel frames have been STRUDL analyzed with tube steel to Richmond connections considered pinned in all directions. These frames were originally analyzed with the joints pinned in two directions, but resisting rotations about the member's axis.

		INSERT AS ONE DIR. FIXED		RT AS NNED
SUPPORT NO.	INSERT INTER.	HILTI BOLT IN.	INSERT INTER.	HILTI BOLT IN.
CC-2-323-112-A43R	0.54	0.27	0.03	0.24
DD-1-016-700-S33R	0.56	N/A	0.45	N/A
FW-1-019-700-C42K	0.44	0.95	0.086	0.85
FW-1-095-700-C62K	0.34	0.74	0.12	0.89
FW-1-096-706-C62K	0.66	N/A	0.62	N/A
FW-1-098-700-C62K	0.22	0.79	0.05	0.86
SF-1-004-700-C46K	0.37	N/A	0.22	N/A

		RT AS R. FIXED	INSERT AS PINNED	
SUPPORT NO.	MAX ₁ Δ	MAX 3	MAX A	MAX, or
CC-2-323-712-A43R	0.021	8179	0.0477	8179
DD-1-016-700-S33R	0.0019	5333	0.002	9918
FW-1-019-700-C42s	0.0042	4752	0.052	6400
FW-1-095-700-C62K	0.0002	5388	0.0004	8423
FW-1-096-706-C62K	0.0028	7702	0.00231	7757
FW-1-098-700-C62K	0.0018	5651	0.0019	5916
SF-1-004-700-C46K	0.032	4950	0.032	4816
	(INCH)	(PSI)	(INCH)	(PSI)