

SHROUD REPAIR PROGRAM
FOR
QUAD CITIES UNIT 2

Shroud Head Contact on Upper Support
GENE-771-113-0695 REV. 0
DRF# B13-01740
Back-up Calculations with Impact Factor for FDDR# EE2-0505
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1. 718H816 Rev. 0 GE Shroud Head and Separators Drawing
2. 112D6541 Rev. 4 GE Upper Support Long Drawing
3. GENE-771-68-1094 Rev. 0 Quad Cities Shroud and Repair Hardware Stress Analysis
4. 919D611P001 Rev. 4 Quad Cities GE Shroud Head Bolt Drawing
5. 112D5487 Rev. 6 Quad Cities GE Reactor, Modification & Installation Drawing
6. GENE-771-69-1094 Rev. 1 Backup Calculations for Quad Cities Shroud Repair Shroud Stress Report

DESCRIPTION

During installation of the shroud head and separators assembly at Quad Cities Unit 2, two of the lifting rod extensions contacted two of the upper supports that are part of the shroud modification assemblies. The purpose of this analysis is to justify the adequacy of the long upper support design to withstand the contact load from the shroud head and separators assembly.

ANALYSIS

According to the sketch provided (Figure 1), the (3"x3.5") cross sectional area of the support bears the highest stresses (bending and shear) due to the contact load (P). According to the references 1 and 4 drawings, the shroud head and separators assembly plus the 48 shroud head bolts weigh $[(126.90 + (48)(0.312))] = 141.9$ kips dry. In this calculation, it is assumed that the total dry weight of the assembly impacted one of the long upper supports at 103° and 283° azimuth locations. Using an impact factor of 1.25, the total load on the support is conservatively assumed to be $(141.9 \times 1.25) = 177.40$ kips.

According to the reference 2 drawing and the Figure 1 AUTOCAD sketch:

MOMENT ARM	= 1.70 in (distance from reactive force)
SHEAR AREA	= (3.0) (3.5) = 10.5 in ²
I (MOM. OF IN.)	= (bh ³ /12) = [(3.5) (3.0) ³]/12 = 7.88 in ⁴
C (TO N-A)	= (h/2) = (3.0/2) = 1.5 in

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Therefore,

$$\text{SHEAR STRESS} = (177.4)/(10.5) = 16.9 \text{ ksi} < .69 S_y = (.6)(47.5) = 28.5 \text{ ksi}$$

$$\text{BENDING STRESS} = [(177.4) (1.7) (1.5)]/(7.9) = 57.3 \text{ ksi} < 1.5 S_m = 71.3 \text{ ksi}$$

The upper support material is X-750 (reference 2) and the value of $S_m = 47.5$ ksi is the minimum value of S_m at 550 °F per paragraph 3.3 of the reference 3 analysis. The 1.5 S_m limit is also extracted from paragraph 5 of the reference 3 analysis. The .6 S_m limit is from ASME Section III Subsection NB-3227.2, 1989 Edition.

Since this conservative calculation shows the maximum stresses to be below their allowables, it is concluded that the actual loads on the two supports did not damage the components.

Note that due to the nature of the shroud modification design (reference 5), the shroud head and separator assembly load on top of the long upper support could not be transferred down into the lower components of the assembly (i.e. the rod, yoke,...) causing axial compressive loading. The shroud head load was reacted by the shroud flange.

Note also that the shroud bearing stress of $[(177.4)/(3.4 \times 3.5)] = 14.91$ ksi is lower than the bearing allowable of $S_y = 30.0$ ksi at 100°F. The bearing stress allowable is from ASME Section III Subsection NG-3227.1, 1989 Edition, and the value of S_y for 304 is from Appendix I of the same ASME code. The dimensions for the bearing area can be found in the reference 5 drawing.

Furthermore, this compressive force on the shroud did not cause an overstress condition in the top guide support plate. According to pages 17 and 18 of the reference 6 report, a compressive load of 165.5 kips by the long upper supports creates a maximum Normal + Upset (OBE) stress of 22.0 ksi. Therefore, the 177.4 kips compressive load on the shroud has created approximately $[(177.4)/(165.5)](22.0) = 23.58$ ksi stress in the top guide support plate. This value is lower than the allowable of $1.5S_m = (1.5)(20.0) = 30.0$ ksi and thus it is acceptable. The S_m value of 20.0 ksi for 304 shroud is from Appendix I of the ASME Section III, Division 1.

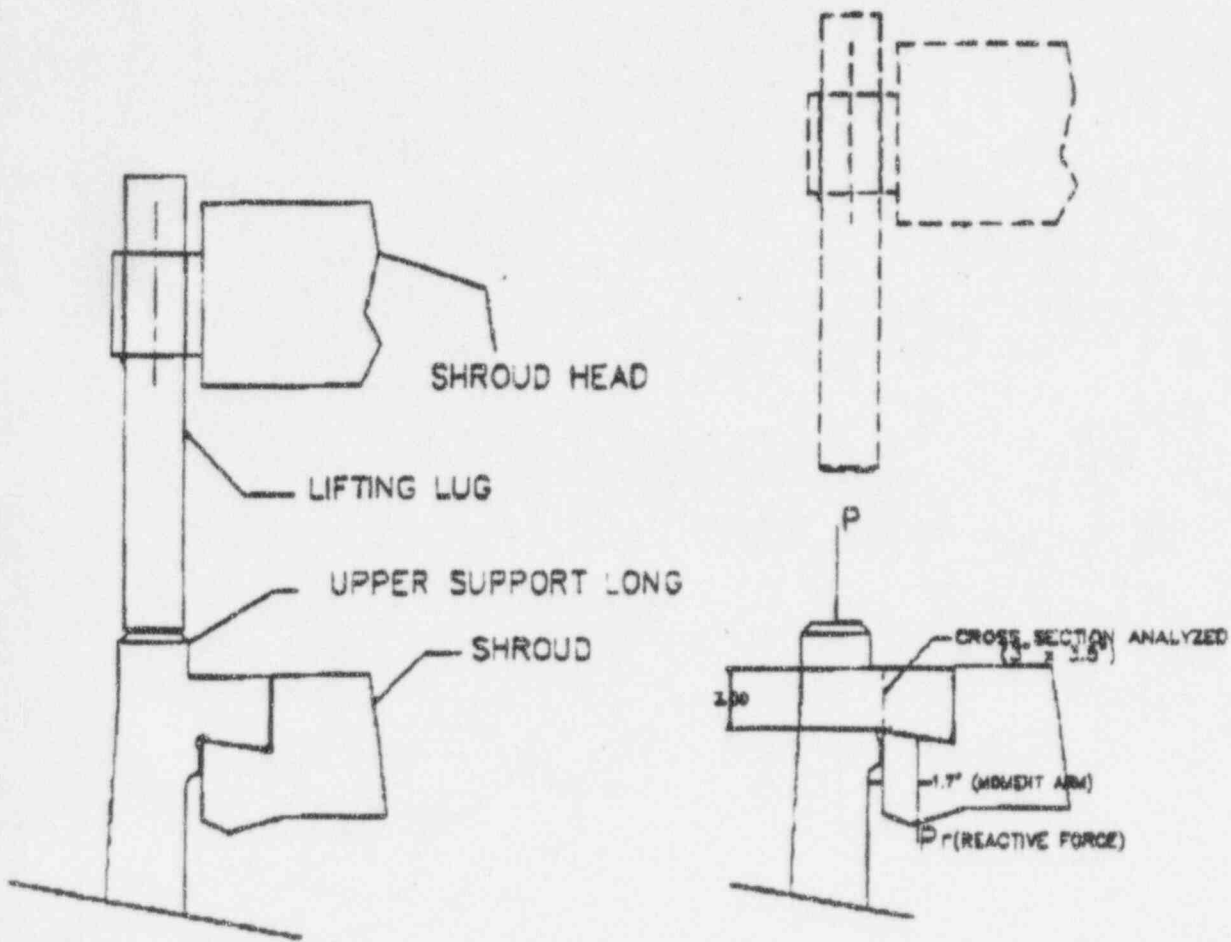


FIGURE 1

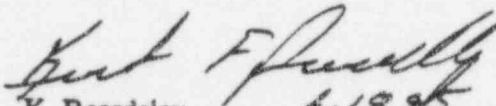
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Criteria Used to determine Impact factor.

References:

1. Whiting Crane Handbook, Whiting Corporation, 1979.
2. Response to Request for Information on the Control of Heavy Loads, Nov. 1981, Rev 1. Section 2.1.7.c no. 13, page 2.1-12. Issued in response to Generic Letter 81-07.
3. AISC Code Section 1.3.3

Reference 1 and 2 provide justification for the use of 1/2% impact for each ft per minute of crane speed with a minimum of 15%. The maximum crane speed for the reactor building 125 Ton crane is 5 feet per minute. This would result in an impact factor of 2.5%. Reference 3 also recommends the use of 25% as the impact factor for traveling cranes and supports. Hence the use of 25% in the analysis for affect on the core shroud repair hardware is conservative as these values provide additional margin above required loads.

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