Georgia Power Company 333 Piedmont Avenue Atlanta, Georgia 30308 Telephone 404 526 7895

Mailing Address: Post Office Box 4545 Atlanta, Georgia 30302

J. C. Causey Vice President Fossil and Hydro Generation



NED-83-237

April 4, 1983

Director of Nuclear Reactor Regulation Attention: Mr. John F. Stolz, Chief Operating Reactors Branch No. 4 Division of Licensing U. S. Nuclear Regulatory Commission Washington, D. C. 20555

# NRC DOCKETS 50-321, 50-366 OPERATING LICENSES DPR-57, NPF-5 EDWIN I. HATCH NUCLEAR PLANT UNITS 1, 2 NUREG-0737 ITEM II.D.1 SAFETY RELIEF VALVE (SRV) TESTING

#### Gentlemen:

Your letter dated December 29, 1982 requested Georgia Power Company (GPC) to provide additional information related to the SRV test program. The attached report provides responses to the six questions from your letter pertaining to the subject item.

Note that question 2 of the response requires a dead weight analysis of the Hatch SRV discharge lines to confirm that results of the test program apply to Plant Hatch. This analysis is being performed for GPC by Bechtel Power Corporation to meet the requirements of I&E Bulletin 79-14 and the Mark I Long-Term Plan. It is anticipated that we will receive a report of this analysis by July 29, 1983, and upon receipt, the report will be submitted to NRC as a supplementary response.

If you require additional information regarding this response, please contact this office.

Very truly yours,

auser C. Causey

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CT/mb

Enclosure

xc: J. T. Beckham, Jr. H. C. Nix, Jr. J. P. O'Reilly (NRC- Region II) Senior Resident Inspector E. I. HATCH NUCLEAR PLANT

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UNIT I

RESPONSES TO

NRC QUESTIONS RELATIVE TO

SRV TESTING

The test program utilized a "rams head" discharge pipe configuration. E. I. Hatch utilizes a "tee" quencher configuration at the end of the discharge line. Describe the discharge pipe configuration used at E. I. Hatch and compare the anticipated loads on valve internals in the E. I. Hatch configuration to the measured loads in the test program. Discuss the impact of any differences in loads on valve operability.

# **RESPONSE TO QUESTION 1**

The safety/relief valve discharge piping configuration at E. I. Hatch utilizes a "tee" quencher at the discharge pipe exit. The average length of the 11 SRV discharge lines (SRVDL) is 107'-9 11/16" and the submergence length in the suppression pool is approximately 7'-8". The SRV test program utilized a ramshead at the discharge pipe exit, a pipe length of 112' and a submergence length of approximately 13'. Loads on valve internals during the test program are larger than loads on valve internals in the E. I. Hatch configuration for the following reasons:

- No dynamic mechanical load originating at the "tee" quencher is transmitted to the valve in the E. I. Hatch configuration because there is at least one anchor point between the valve and the tee quencher.
- 2. The first length of the segment of piping downstream of the SRV in the test facility was longer than the E. I. Hatch piping, thereby resulting in a bounding dynamic mechanical load on the valve in the test program due to the larger moment arm between the SRV and the first elbow. The first segment length in the test facility is 12 ft. whereas this length in the E. I. Hatch configuration is given below.

Vent	Length	Vent	Length						
A	12' - 11 3/8"*	F	2' - 5 5/8"						
В	3' - 4 <sup>1</sup> z''	G	7' - 4 5/8"						
С	7' - 10 5/8"	Е	12' - 11 5/8"*						
D	1' - 5 5/8"	J	2' - 8 <sup>15</sup>						
Е	1' - 3"	K	2' - 10 5/8"						
		L	7' - 9"						

\*The first segment length in the test facility does not include the elbow length. The above first segment length for the vents include the elbow length. Therefore by subtracting the elbow length from vents A and H all the first segment lengths would be less than 12 feet.

- 3. Dynamic hydraulic loads (backpressure) are experienced by the valve internals in the E. I. Hatch configuration. The backpressure loads may be either (i) transient backpressures occurring during valve actuation, or (ii) steady-state backpressures occurring during steady-state flow following valve actuation.
  - (a) The key parameters affecting the transient backpressures are the fluid pressure upstream of the valve, the valve opening time, the fluid inertia in the submerged SRVDL and the SRVDL air volume. Transient backpressures increase with higher upstream pressure, shorter valve opening times, greater line submergence, and smaller SRVDL air volume. The transient backpressure in the test program was maximized by utilizing a submergence of 13', and a pipe length of 112'. The maximum transient backpressure occurs with high pressure steam flow conditions. The transient backpressure for the alternate shutdown cooling mode of operation is always much less than the design for steam flow conditions because of the lower upstream pressure and the longer valve opening time.

(b) The steady-state backpressure in the test program was maximized by utilizing an orifice plate in the SRVDL above the water level and before the ramshead. The orifice was sized to produce a backpressure greater than that calculated for any of the E. I. Hatch SRVDL's.

The differences in the line configuration between the E. I. Hatch plant and the test program as discussed above result in the loads on the valve internals for the test facility which bound the actual E. I. Hatch loads. An additional consideration in the selection of the ramshead for the test facility was to allow more direct measurement of the thrust load in the final pipe segment. Utilization of a "tee" quencher in the test program would have required quencher supports that would unnecessarily obscure accurate measurement of the pipe thrust loads. For the reasons stated above, differences between the SRVDL configurations in the E. I. Hatch and the test facility will not have any adverse effect on SRV operability at E. I. Hatch relative to the test facility.

The test configuration utilized no spring hangers as pipe supports. Plant specific configurations do use spring hangers in conjunction with snubber and rigid supports. Describe the safety relief valve pipe supports used at E. I. Hatch and compare the anticipated loads on valve internals for the E. I. Hatch pipe supports to the measured loads in the test program. Describe the impact of any differences in loads on valve operability.

# **RESPONSE TO QUESTION 2**

The E. I. Hatch safety-relief valve discharge lines (SRVDL's) are supported by a combination of snubbers, rigid supports, and spring hangers. The locations of snubbers and rigid supports at E. I. Hatch are such that the location of such supports in the BWR generic test facility is prototypical, i.e., in each case (E. I. Hatch and the test facility) there are supports near each change of direction in the pipe routing. Additionally, each SRVDL at E. I. Hatch has only 1 to 2 spring hangers, all of which are located in the drywell. The spring hangers, snubbers, and rigid supports were designed to accommodate combinations of loads resulting from piping dead weight, thermal conditions, seismic and suppression pool hydrodynamic events, and a high pressure steam discharge transient.

The dynamic load effects on the piping and supports of the test facility due to the water discharge event (the alternate shutdown cooling mode) were found to be significantly lower than corresponding loads resulting from the high pressure steam discharge event. As stated in NEDE-24988-P, this finding is considered generic to all BWR's since the test facility was designed to be prototypical of the features pertinent to this issue. Furthermore, analysis of the E. I. Hatch SRVDL configurations will be performed to confirm the applicability of this conclusion to E. I. Hatch.

During the water discharge transient there will be significantly lower dynamic loads acting on the snubbers and rigid supports than during the steam discharge transient. This will more than offset the small increase in the dead load on these supports due to the weight of the water during the alternate shutdown cooling mode of operation. Therefore, design adequacy of the snubbers and rigid supports is assured as they are designed for the larger steam discharge transient loads.

This question addresses the design adequacy of the spring hat with respect to the increased dead load due to the weight of the water during the liquid discharge transient. As was discussed with respect to snubbers and rigid supports, the dynamic loads resulting from liquid discharge during the alternate shutdown cooling mode of operation are significantly lower than those from the high pressure steam discharge. Therefore, it is believed that sufficient margin exists in the E. I. Hatch piping system design to adequately offset the increased dead load on the spring hangers in an unpinned condition due to a water filled condition. Furthermore, the effect of the water dead weight load does not affect the ability of SRVs to open to establish the alternate shutdown cooling path since the loads occur in the SRVDL only after valve opening.

Report NEDE-24988-P did not identify any valve functional deficiencies or anomalies encountered during the test program. Describe the impact on valve safety function of any valve functional deficiencies or anomalies encountered during the program.

# **RESPONSE TO QUESTION 3**

No functional deficiencies or anomalies of the safety relief or relief valves were experienced during the testing at Wyle Laboratories for compliance with the alternate shutdown cooling mode requirement. All of the valves subjected to test runs, valid and invalid, opened and closed without loss of pressure integrity or damage. Anomalies encountered during the test program were all due to failures of test facility instrumentation, equipment, data acquisition equipment, or deviation from the approved test procedure.

The test specification for each valve required six runs. Under the test procedure, any anomaly caused the test run to be judged invalid. All anomalies were reported in the test report. The Wyle Laboratories test log sheet for the Target Rock Model 7567F valve tests is attached. This valve is used in the E. I. Hatch Nuclear Power Station.

Each Wyle test report for the respective values identifies each test run performed and documents whether or not the test run is valid or invalid and states the reason for considering the run invalid. No anomaly encountered during the required test program affects any value safety or operability function.

All valid test runs are identified in Table 2.2-1 of NEDE-24988-P. The data presented in Table 4.2-1 for each valve were obtained from the Table 2.2-1 test runs and were based upon the selection criteria of:

(a) Presenting the maximum representative loading information obtained from the steam run data,

- (b) Presenting the maximum representative water loading information obtained from the 15°F subcooled water test data,
- (c) Presenting the data on the only test run performed for the 50°F subcooled water test condition.

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	TESI REPORT	REPORT NO 17476-04
SCIENTIFIC SERVICES AND SYSTEMS GROUP	• •	OUR JCB NO. 17476
General Electric Company 175 Curtner Avenue San Jose, California	ר ר	CONTRACT N/A PAGE 1 of PAGE REPC
L	μ.	DATE May 8, 1981 Revision A: January 18, 1

#### 1.0 PURPOSE

The purpose of this report is to present the requirements, procedures, and results of steam and low pressure water operability tests performed on a Target Rock 6X10 Safety Relief Valve (SRV) identified as TR-1. The tests were performed to determine if the SRV would operate properly when subjected to the test conditions specified in General Electric Specification 22A7424, Revision B.

# 2.0 REFERENCES

- 2.1 General Electric Purchase Order 205-XH212.
- 2.2 Wyle Laboratories' Test Procedure No. 17450-15
- 2.3 Wyle Laboratories' Test Procedure No. 17450-01.
- 2.4 General Electric Specification 22A7424, Revision B.
- 2.5 Wyle Laboratories' Report No. 17450-02.
- 2.6 Wyle Laboratories' Test Report No. 45503-04.
- 2.7 Wyle Laboratories Test Report No. 45503-07.
- 3.0 MANUFACTURER

Target Rock Corporation 1966E Broadhollow Road East Farmingdale, NY 11735

STATE OF ALABAMA	Whe shak have no kapitry for damages of any kind to person or property including specie considurencial damages, resulting from this s providing the services covered by this rep
COUNTY OF MADISON 1	STEAN SEDVICES
Larry E. Frazier being duly swom	TEST BY STEAM SERVICES
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Notary Public in and for the State of Alabama at large.	
Wy Commission expires 1 nov. 30 19 85	

# PAGE NO. 8

TEST REPORT NO. 17476-04

# TABLE 1

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TEST LCG FOR SRV TR-1

Test No.	Test Media	Load Line Configuration	Test Date	Remarks
301	Steam	1	3/17/81	Acceptable
302	Water	1	3/17/81	GN, Regulator failed. Data not acceptable.
30)	Water	1	3/17/81	Acceptable
304	Steam	1	3/17/81	Acceptable
305	Water	- 1	3/18/81	Acceptable
306	Steam	1	3/18/81	Acceptable
307	Water	. 1	3/18/81	Acceptable
308	Water	· 1	3/18/81	Special test at elevated temperature and low pres- sure requested by G.E.

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The purpose of the test program was to determine valve performance under conditions anticipated to be encountered in the plants. Describe the events and anticipated conditions at E. I. Hatch for which the valves are required to operate and compare these plant conditions to the conditions in the test program. Describe the plant features assumed in the event evaluations used to scope the test program and compare them to plant features at E. I. Hatch. For example, describe high level trips to prevent water from entering the steam lines under high pressure operating conditions as assumed in the test event and compare them to trips used at E. I. Hatch.

# RESPONSE TO NRC QUESTION 4

The purpose of the S/RV test program was to demonstrate that the Safety Relief Valve (S/RV) will open and reclose under all expected flow conditions. The expected valve operating conditions were determined through the use of analyses of accidents and anticipated operational occurrences referenced in Regulatory Guide 1.70, Revision 2. Single failures were applied to these analyses so that the dynamic forces on the safety and relief valves would be maximized. Test pressures were the highest predicted by conventional safety analysis procedures. The BWR Owners Group, in their enclosure to the September 17, 1980 letter from D. B. Waters to R. H. vollmer, identified 13 events which may result in liquid or two-phase S/RV inlet flow that would maximize the dynamic forces on the safety and relief valve. These events were identified by evaluating the initial events described in Regulatory Guide 1.70, Revision 2, with and without the additional conservatism of a single active component failure or operator error postulated in the event sequence. It was concluded from this evaluation that the alternate shutdown cooling mode is the only expected event which will result in liquid at the valve inlet. Consequently, this was the event simulated in the S/RV test program. This conclusion and the test results applicable to E. I. Hatch are discussed below. The alternate shutdown cooling mode of operation has been described in the response to NRC Question 5.

The S/RV inlet fluid conditions tested in the BWR Owners Group S/RV test program, as documented in NEDE-24988-P, are 15° to 50° subcooled liquid at 20 psig to 250 psig. These fluid conditions envelope the conditions expected to occur at E. I. Hatch in the alternate shutdown cooling mode of operation.

The BWR Owners Group identified 13 events by evaluating the initiating events described in Regulatory Guide 1.70, Revision 2, with the additional conservatism of a single active component failure or operator error postulated in the events sequence These events and the plant-specific features that mitigate these events, are summarized in Table 1. Of these 13 events, only 10 are applicable to the E. I. Hatch plant because of its design and specific plant configuration. 3 events, namely 5, 6, & 10 are not applicable to the E. I. Hatch plant for the reasons listed below:

- (a) Events 5 and 10 are not applicable, because Plant E. I. Hatch does not have a HPCS system.
- (b) Event 6 is not applicable because Plant E. I. Hatch does not have RCIC head sprays.

For the 10 remaining events, the E. I. Hatch specific features, such as trip logic, power supplies, instrument line configuration, alarms and operator actions, have been compared to the base case analysis presented in the BWR Owners Group submittal of September 17, 1980. The comparison has demonstrated that in each case, the base case analysis is applicable to E. I. Hatch because the base case analysis does not include any plant features which are not already present in the E. I. Hatch design. For these events, Table 1 demonstrates that the E. I. Hatch specific features are included in the base case analyses presented in the BWR Owners Group submittal of September 17, 1980. It is seen from Table 1, that all plant features assumed in the event evaluation are also existing features in the E. I. Hatch plant. All features included in this base case analysis are similar to plant features in the E. I. Hatch design. Furthermore, the time available for operator action is expected to be longer in the E. I. Hatch plant than in the base case analysis for each case where operator action is required.

Event 7, the alternate shutdown cooling mode of operation, is the only expected event which will result in liquid or two-phase fluid at the S/RV inlet. Consequently, this event was simulated in the BWR S/RV test program. In E. I. Hatch, this event involves flow of subcooled water (approximately 130°F subcooled) at a pressure of approximately 85 psig. The test conditions clearly envelope these plant conditions.

As discussed above, the BWR Owners Group evaluated transients including single active failures that would maximize the dynamic forces on the safety relief valves. As a result of this evaluation, the alternate shutdown cooling mode is the only expected event involving liquid or two-phase flow. Consequently this event was tested in the BWR S/RV test program. The fluid conditions and flow conditions tested in the BWR Owners Group test program conservatively envelope the E. I. Hatch plantspecific fluid conditions expected for the alternate shutdown cooling mode of operation.

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							1 -	1	7	
#13 LEA, ECCS Overf Brk	×	×	~						_	•
#12 SSA, Depress. & ECCS Over., Operator Error	×	2								
#11 SBA, HPCI, HPCI L8 Trip Failure	•			.  ×	1				·	
#10 SBA, HPCS, HPCS L8 Trip Failure								·	_	
#9 SSA, RCIC, RCIC L8 Trip Failure					X				_	
#8 WZL BFK OSC							× ×	××	N	
<pre>#7 Alt. Shutdown Cooling, Shutdown Suction Unstailsble</pre>				-						
#6 Transient RCIC Hd. Spr.										
#5 Transient HPCS, HPCS L8 Trip Failure				-					<u> </u>	.:.
#4 Transient RCIC, RCIC L8 Trip Failure					×	S			1	
#3 Transient HPCI, HPCI L8 Trip Failure					X					
#2 Press. Reg. Fail.	•					X	X	×		
#1 FW Cont. Fail., FW L8 Trip Failure				X		×	X			
ZERUTAER THAIR		Pressure ECCS Initiation on igh Drywell Pressure	Pressure Initiation on Low ater Level	<sup>oumps</sup> Trip on Low Suction ressure	t Trip on Nigh Backpressure	C Trip on High Backpressure	Vs Closure on Low Turbine nlet Pressure	Vs Closure on High Steam Flow	Vs Closure on High Steam Tunnel emperature	•
		Low	Lou	FW.	IIPO	RC	HSI I	HS	HS	ļ

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PLANT FEATURES	ter Level 7 Alarm	ywell Pressure Alarm X	vel 8 Trip	vel 8 Trip	and RCIC Initiation on Alater Level	Initiation on High Drywell sure	nitiation on High Drywell sure
FW L8 Trip Failure #2 Press. Reg. Fail.	~ ~ ~	N N N			100		
#3 Transient HPCI, HPCI L8 Trip Failure	X		2°	X	S S	s	.
#4 Transient RCIC, RCIC L8 Trip Failure #5 Transient HPCS,	XX	XXX	XXX	X	X S X	S	
#6 Transient RCIC Hd. *	. NA		AN AN	X	A NA		
.prifoon controver Cooling. Shutdown Suction eldsfisvsnU							
#8 HZT BLK 02C				X	S		Ī
#10 SBA, HPCS,	×	×	A A	No No	x x	s	ŀ
HPCS L& Trip Failure #11 SSA, HPCI, #11 SSA, HPCI,	X NA S	×	X AN		×	NA S	
#12 SEA, Depress. & ECCS Over., Operator Error	×			2	×	S	-
#13 LBA, ECCS OVERT BEK	×	× 1	XXX	X	×	×	2 S

#13 LEA, ECCS OVERT Erk	Τ	Τ	Τ			X	x x	X	s.	
#12 SSA, Depress. & ECCS Over., Operator Error						X	s			
#11 SSA, HPCI, HPCI L8 Trip Failure						X	S			
#10 SSA, HPCS, HPCS L8 Trip Failure						X	AN			د.
#9 _ SSA, RCIC, _ RCIC LS Trip Failure						×	s			
*8 HZT 84K 02C		S			X	S		1	_	
.prifoon Cooling, Shutdown Suction SfdsfisvanU			•							•
#6 Transient RCIC Hd. Spr.					·					
#5 Transient HPCS, HPCS LB Trip Failure										
#4 Transient RCIC, RCIC L8 Trip Failure										
#3 Transient, HPCI, HPCI L8 Trip Failure		•								sis
#2 Press. Reg. Fail			×	×	×					Analy
#1 FW Cont. Fail., FW Lont. Failure			X							e Case
ZƏRUTAƏƏ TMAL9		MSIV Closure on High Radiation	Reactor Scram on Turbine Trip	Reactor Scram on Neutron Flux Honitor .	Reactor Scram on MSIVs Closure	Réactor Scram on High Radiation	Reactor Scram on High Drywell Pressure	Reactor Scram on Low Water Level	Reactor Isolation on Low Water Level	v continue considered in Base

KEY: X - Feature consumered S - Feature in Plant Specific Design NA - Not Applicable

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The values are likely to be extensively cycled in a controlled depressurization mode in a plant-specific application. Was this mode simulated in the test program? What is the effect of this value cycling on value performance and probability of the value to fail open or to fail closed?

# **RESPONSE TO NRC QUESTION 5**

The BWR safety/relief valve (SRV) operability test program was designed to simulate the alternate shutdown cooling mode, which is the only expected liquid discharge event for E. I. Hatch. The sequence of events leading to the alternate shutdown cooling mode is given below.

Following normal reactor shutdown, the reactor operator depressurizes the reactor vessel by opening the turbine bypass valves and removing heat through the main condenser. If the main condenser is unavailable, the operator could depressurize the reactor vessel by using the SRV's to discharge steam to the suppression pool. If SRV operation is required, the operator cycles the valves in order to assure that the cooldown rate is maintained within the technical specification limit of 100°F per hour. When the vessel is depressurized, the operator initiates normal shutdown cooling by use of the RHR system. If that system is unavailable because the valve on the RHR shutdown cooling suction line fails to open, the operator initiates the alternate shutdown cooling mode.

For alternate shutdown cooling, the operator opens one SRV and initiates either an RHR or core spray pump utilizing the suppression pool as the suction source. The reactor vessel is filled such that water is allowed to flow into the main steam lines and out of the SRV and back to the suppression pool. Cooling of the system is provided by use of an RHR heat exchanger. As a result, an alternate cooling mode is maintained.

In order to assure continuous long term heat removal, the SRV is kept open and no cycling of the valve is performed. In order to control the reactor vessel cooldown rate, the operator is instructed to control the flow rate into the vessel. Consequently, no cycling of the SRV is required for the alternate shutdown cooling mode, and no cycling of the SRV was performed for the generic BWR SRV operability test program.

The ability of the E. I. Hatch SRV to be extensively cycled for steam discharge conditions has been confirmed during steam discharge qualification testing of the valve by the valve vendor. Based on the qualification testing of the SRV's, the cycling of the valves in a controlled depressurization mode for steam discharge conditions will not adversely affect valve performance and the probability of the valve to fail open or closed is extremely low.

Describe how the values of value  $C_v$ 's in report NEDE-24988-P will be used at E. I. Hatch. Show that the methodology used in the test program to determine the value  $C_v$  will be consistent with the application at E. I Hatch.

# RESPONSE TO NRC QUESTION 6

The flow coefficient,  $C_v$ , for the Target Rock Model 7567F safety relief valve (SRV) utilized in E. I. Hatch was determined in the generic SRV test program (NEDE-24988-P). The average flow coefficient calculated from the test results for the Target Rock Model 7567F, is reported in Table 5.2-1 of NEDE-24988-P. This test value has been used by Georgia Power to confirm that the liquid discharge flow capacity of the E. I. Hatch SRV's will be sufficient to remove core decay heat when injecting into the reactor pressure vessel (RPV) in the alternate shutdown cooling mode. The C<sub>v</sub> value determined in the SRV test demonstrates that the E. I. Hatch SRV's are capable of returning the flow injected by the RER or CS pump to the suppression pool.

If it were necessary for the operator to place the E. I. Hatch plant in the alternate shutdown cooling mode, he would assure that adequate core cooling was being provided by monitoring the following parameters: RHR or CS flow rate, reactor vessel pressure and reactor vessel temperature.

The flow coefficient for the Target Rock Model 7567F value reported in NEDE-24988-P was determined from the SRV flow rate when the value inlet was pressurized to approximately 250 psig. The value flow rate was measured with the supply line flow venturi upstream of the steam chest. The  $C_v$  for the value was calculated using the nominal measured pressure differential between the value inlet (steam chest) and 3' downstream of the value and the corresonding measured flowrate. Furthermore, the test conditions and test configuration were representative of E. I. Hatch plant conditions for the alternate shutdown cooling mode, e.g. pressure upstream of the value, fluid temperature, friction losses and liquid flowrate. Therefore, the reported  $C_v$  values are appropriate for application to the E. I. Eatch plant.

# REFERENCES

- F. L. Leverenz, "Probabilistic Evaluation of High Pressure Liquid Challenge of Safety/Relief Valve Piping," Science Applications, Inc. Palo Alto, California, April 1981.
- Letter to D. G. Eisenhut (USNRC) from T. D. Keenan (BWR Owners Group), November 14, 1979.
- Letter to R. H. Vollmer (USNRC) from D. B. Waters (BWR Owners Group), September 17, 1980.

E. I. HATCH NUCLEAR PLANT

UNIT 2

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RESPONSES TO

NRC QUESTIONS RELATIVE TO

SRV TESTING

The test program utilized a "ramshead" discharge pipe configuration. E. I. Hatch utilizes a "tee" quencher configuration at the end of the discharge line. Describe the discharge pipe configuration used at E. I. Hatch and compare the anticipated loads on valve internals in the E. I. Hatch configuration to the measured loads in the test program. Discuss the impact of any differences in loads on valve operability.

# RESPONSE TO QUESTION 1

The safety/relief valve discharge piping configuration at E. I. Hatch utilizes a "tee" quencher at the discharge pipe exit. The average length of the 11 SRV discharge lines (SRVDL) is 109'-2 7/8" and the submergence length in the suppression pool is approximately 7'-8". The SRV test program utilized a ramshead at the discharge pipe exit, a pipe length of 112' and a submergence length of approximately 13'. Loads on valve internals during the test program are larger than loads on valve internals in the E. I Hatch configuration for the following reasons:

- No dynamic mechanical load originating at the "tee" quencher is transmitted to the valve in the E. I. Hatch configuration because there is at least one anchor point between the valve and the tee quencher.
- 2. The first length of the segment of piping downstream of the SRV in the test facility was longer than the E. I. Hatch piping, thereby resulting in a bounding dynamic mechanical load on the valve in the test program due to the larger moment arm between the SRV and the first elbow. The first segment length in the test facility is 12 ft. whereas this length in the plant configuration is given below:

Vent	"A"	-	19 5/8"		Vent	"E"	-	19 5/8"	Vent	"K"	-	4'0
Vent	В	-	2'-3"	•	Vent	F	-	4'-0	Vent	L	-	2.234"
Vent	С	-	4'-0		Vent	G	-	4'-0	Vent	М	-	221/2"
Vent	D.	-	5'-10"		Vent	н	-	19 5/8"				

- 3. Dynamic hydraulic loads (backpressure) are experienced by the valve internals in the E. I. Hatch configuration. The backpressure loads may be either (i) transient backpressures occurring during valve actuation, or (ii) steady-state backpressures occurring during steady-state flow following valve actuation.
  - (a) The key parameters affecting the transient backpressures are the fluid pressure upstream of the valve, the valve opening time, the fluid inertia in the submerged SRVDL and the SRVDL air volume. Transient backpressures increase with higher upstream pressure, shorter valve opening times, greater line submergence, and smaller SRVDL air volume. The transient backpressure in the test program was maximized by utilizing a submergence of 13', and a pipe length of 112'. The maximum transient backpressure occurs with high pressure steam flow conditions. The transient backpressure for the alternate shutdown cooling mode of operation is always much less than the design for steam flow conditions because of the lower upstream pressure and the longer valve opening time.
  - (b) The steady-state backpressure in the test program was maximized by utilizing an orifice plate in the SRVDL above the water level and before the ramshead. The orifice was sized to produce a backpressure greater than that calculated for any of the SRVDL's.

The differences in the line configuration between the E. I. Hatch plant and the test program as discussed above result in the loads on the valve internals for the test facility which bound the actual E. I. Hatch loads. An additional consideration in the selection of the ramshead for the test facility was to allow more direct measurement of the thrust load in the final pipe segment. Utilization of a "tee" quencher in the test program would have required quencher supports that would unnecessarily obscure accurate measurement of the pipe thrust loads. For the reasons stated above, differences between the SRVDL configurations in E. I. Hatch and the test facility will not have any adverse effect on SRV operability at E. I. Hatch relative to the test facility.

The test configuration utilized no spring hangers as pipe supports. Plant specific configurations do use spring hangers in conjunction with snubber and rigid supports. Describe the safety relief valve pipe supports used at E. I. Hatch and compare the anticipated loads on valve internals for the E. I. Hatch pipe supports to the measured loads in the test program. Describe the impact of any differences in loads on valve operability.

# **RESPONSE TO QUESTION 2**

The E. I. Hatch safety-relief valve discharge lines (SRVDL's) are supported by a combination of snubbers, rigid supports, and spring hangers. The locations of snubbers and rigid supports at E. I. Hatch are such that the location of such supports in the BWR generic test facility is prototypical, i.e., in each case (E. I. Hatch and the test facility) there are supports near each change of direction in the pipe routing. Additionally, each SRVDL at E. I. Hatch has only 1 to 2 spring hangers, all of which are located in the drywell. The spring hangers, snubbers, and rigid supports were designed to accommodate combinations of loads resulting from piping dead weight, thermal conditions, seismic and suppression pool hydrodynamic events, and a high pressure steam discharge transient.

The dynamic load effects on the piping and supports of the test facility due to the water discharge event (the alternate shutdown cooling mode) were found to be significantly lower than corresponding loads resulting from the high pressure steam discharge event. As stated in NEDE-24988-P, this finding is considered generic to all EWR's since the test facility was designed to be prototypical of the features pertinent to this issue. Furthermore, analysis of the E. I. Hatch SRVDL configurations will be performed to confirm the applicability of this conclusion to E. I. Hatch.

During the water discharge transient there will be significantly lower dynamic loads acting on the snubbers and rigid supports than during the steam discharge transient. This will more than offset the small increase in the dead load on these supports due to the weight of the water during the alternate shutdown cooling mode of operation. Therefore, design adequacy of the snubbers and rigid supports is assured as they are designed for the larger steam discharge transient loads.

This question addresses the design adequacy of the spring hangers with respect to the increased dead load due to the weight of the water during the liquid discharge transient. As was discussed with respect to snubbers and rigid supports, the dynamic loads resulting from liquid discharge during the alternate shutdown cooling mode of operation are significantly lower than those from the high pressure steam discharge. Therefore, it is believed that sufficient margin exists in the E. I. Hatch piping system design to adequately offset the increased dead load on the spring hangers in an unpinned condition due to a water filled condition. Furthermore, the effect of the water dead weight load does not affect the ability of SRVs to open to establish the alternate shutdown cooling path since the loads occur in the SRVDL only after valve opening.

Report NEDE-24988-P did not identify any valve functional deficiencies or anomalies encountered during the test program. Describe the impact on valve safety function of any valve functional deficiencies or anomalies encountered during the program.

#### **RESPONSE TO QUESTION 3**

No functional deficiencies or anomalies of the safety relief or relief valves were experienced during the testing at Wyle Laboratories for compliance with the alternate shutdown cooling mode requirement. All of the valves subjected to test runs, valid and invalid, opened and closed without loss of pressure integrity or damage. Anomalies encountered during the test program were all due to failures of test facility instrumentation, equipment, data acquisition equipment, or deviation from the approved test procedure.

The test specification for each valve required six runs. Under the test procedure, any anomaly caused the test run to be judged invalid. All anomalies were reported in the test report. The Wyle Laboratories test log sheet for the Target Rock Model 7567F valve tests is attached. This valve is used in the E. I. Hatch Nuclear Power Station.

Each Wyle test report for the respective values identifies each test run performed and documents whether or not the test run is valid or invalid and states the reason for considering the run invalid. No anomaly encountered during the required test program affects any value safety or operability function.

All valid test runs are identified in Table 2.2-1 of NEDE-24988-P. The data presented in Table 4.2-1 for each valve were obtained from the Table 2.2-1 test runs and were based upon the selection criteria of:

(a) Presenting the maximum representative loading information obtained from the steam run data.

- (b) Presenting the maximum representative water loading information obtained from the 15°F subcooled water test data.
- (c) Presenting the data on the only test run performed for the 50°F subcooled water test condition.

· · · ·	TESI REPORT	REPORT NO_ 17476-04
SCIENTIFIC SERVICES AND SYSTEMS GROUP		OUR JCB NO. 17476
General Electric Company 175 Curtner Avenue San Jose, California	<b>٦</b> .	CONTRACT N/A PAGE 1 of PAGE REPO
L	. ۲	DATE May 8, 1981 Revision A: January 18, 1

#### 1.0 PURPOSE

The purpose of this report is to present the requirements, procedures, and results of steam and low pressure water operability tests performed on a Target Rock 6X10 Safety Relief Valve (SRV) identified as TR-1. The tests were performed to determine if the SRV would operate properly when subjected to the test conditions specified in General Electric Specification 22A7424, Revision B.

# 2.0 REFERENCES

- 2.1 General Electric Purchase Order 205-XH212.
- 2.2 Wyle Laboratories' Test Procedure No. 17450-15
- 2.3 Wyle Laboratories' Test Procedure No. 17450-01.
- 2.4 General Electric Specification 22A7424, Revision B.
- 2.5 Wyle Laboratories' Report No. 17450-02.
- 2.6 Wyle Laboratories' Test Report No. 45503-04.
- 2.7 Wyle Laboratories Test Report No. 45503-07.
- 3.0 MANUFACTURER

Target Rock Corporation 1966E Broadhollow Road East Farmingdale, NY 11735

STATE OF ALABAMA	Wwe shak have no sability for camages of any kind to person or property including special consequencial camages, resulting from Wwe's providing the services covered by this repo
Larry E. Frazier	TEST BY STEAM SERVICES
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TEST REPORT NO. 17476-04

# TABLE I

TEST LCG FOR SRV TR-1

Test No.	Test Hedla	Load Line Configuration	Test Date	Remarks
301	Steam	1	3/17/81	Acceptable
302	Water	1	3/17/81	GN, Regulator failed. Data not acceptable.
303	Water	1	3/17/81	Acceptable
304	Steam	1	3/17/81	Acceptable
305	Water	. 1	3/18/81	Acceptable
306	Steam	1	3/18/81	Acceptable
307	Water	. 1	3/18/81	Acceptable
308	Water	· · · ·	3/18/81	Special test at elevated temperature and low pres- sure requested by G.E.

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# WYLE LAEORATORIES

The purpose of the test program was to determine valve performance under conditions anticipated to be encountered in the plants. Describe the events and anticipated conditions at E. I. Hatch for which the valves are required to operate and compare these plant conditions to the conditions in the test program. Describe the plant features assumed in the event evaluations used to scope the test program and compare them to plant features at E. I. Hatch. For example, describe high level trips to prevent water from entering the steam lines under high pressure operating conditions as assumed in the test event and compare them to trips used at E. I. Hatch.

#### **RESPONSE TO NRC QUESTION 4**

The purpose of the S/RV test program was to demonstrate that the Safety Relief Valve (S/RV) will open and reclose under all expected flow conditions. The expected valve operating conditions were determined through the use of analyses of accidents and anticipated operational occurrences referenced in Regulatory Guide 1.70, Revision 2. Single failures were applied to these analyses so that the dynamic forces on the safety and relief valves would be maximized. Test pressures were the highest predicted by conventional safety analysis procedures. The BWR Owners Group, in their enclosure to the September 17, 1980 letter from D. B. Waters to R. H. Vollmer, identified 13 events which may result in liquid or two-phase S/RV inlet flow that would maximize the dynamic forces on the safety and relief valve. These events were identified by evaluating the initial events described in Regulatory Guide 1.70, Revision 2, with and without the additional conservatism of a single active component failure or operator error postulated in the event sequence. It was concluded from this evaluation that the alternate shutdown cooling mode is the only expected event which will result in liquid at the valve inlet. Consequently, this was the event simulated in the S/RV test program. This conclusion and the test results applicable to E. I. Hatch are discussed below. The alternate shutdown cooling mode of operation has been described in the response to NRC Question 5.

The S/RV inlet fluid conditions tested in the BWR Owners Group S/RV test program, as documented in NEDE-24988-P, are 15° to 50° subcooled liquid at 20 psig to 250 psig. These fluid conditions envelope the conditions expected to occur at E. I. Hatch in the alternate shutdown cooling mode of operation.

The BWR Owners Group identified 13 events by evaluating the initiating events described in Regulatory Guide 1.70, Revision 2, with the additional conservatism of a single active component failure or operator error postulated in the events sequence. These events and the plant-specific features that mitigate these events, are summarized in Table 1. Of these 13 events, only 10 are applicable to the E. I. Hatch plant because of its design and specific plant configuration. 3 events, namely 5, 6, & 10 are not applicable to the E. I. Hatch plant for the reasons listed below:

- (a) Events 5 and 10 are not applicable, because Plant E. I. Hatch does not have a HPCS system.
- (b) Event 6 is not applicable because Plant E. I. Hatch does not have RCIC head sprays.

For the 10 remaining events, the E. I. Hatch specific features, such as trip logic, power supplies, instrument line configuration, alarms and operator actions, have been compared to the base case analysis presented in the BWR Owners Group submittal of September 17, 1980. The comparison has demonstrated that in each case, the base case analysis is applicable to E. I. Hatch because the base case analysis does not include any plant features which are not already present in the E. I. Hatch design. For these events, Table 1 demonstrates that the E. I. Hatch specific features are included in the base case analyses presented in the BWR Owners Group submittal of September 17, 1980. It is seen from Table 1, that all plant features assumed in the event evaluation are also existing features in the E. I. Hatch plant. All features included in this base case analysis are similar to plant features in the E. I. Hatch design. Furthermore, the time available for operator action is expected to be longer in the E. I. Hatch plant than in the base case analysis for each case where operator action is required.

Event 7, the alternate shutdown cooling mode of operation, is the only expected event which will result in liquid or two-phase fluid at the S/RV inlet. Consequently, this event was simulated in the BWR S/RV test program. In E. I. Hatch, this event involves flow of subcooled water (approximately 130°F subcooled) at a pressure of approximately 85 psig. The test conditions clearly envelope these plant conditions.

As discussed above, the BWR Owners Group evaluated transients including single active failures that would maximize the dynamic forces on the safety relief valves. As a result of this evaluation, the alternate shutdown cooling mode is the only expected event involving liquid or two-phase flow. Consequently this event was tested in the BWR S/RV test program. The fluid conditions and flow conditions tested in the BWR Owners Group test program conservatively envelope the E. I. Hatch plantspecific fluid conditions expected for the alternate shutdown cooling mode of operation.

					and the second				1 -		
#13 LBA, ECCS Overf Erk		X	X	03							_
#12 SSA, Depress. & ECCS Over., Operator Error		×	2								
#11 SSA, HPCI, HPCI LS Trip Failure						1					
#10 SSA, HPCS, HPCS L8 Trip Failure											
#9 SBA, RCIC, RCIC L8 Trip Failure						X	S				
HE HEL BEK OSC		1					1		S	× ×	S
.pnifool nyobtuda. JfA 74 Shutdown Suction Unsvsilsble	ŧ				1-						
6 Transient RCIC Hd. Spr.	4										
5 Transient HPCS, HPCS L8 Trip Failure	#				_						
4 Transient RCIC, RCIC LB Trip Failure	#						×				
3 Transient HPCI, HPCI L8 Trip Failure	:#					×					:
? Press. Reg. Fail.	:#	•		•			ſ	×	X	×	
L FW Cont. Fail., FW L8 Trip Failure	Ľ#				×			×	×		
		•	tiation on re	on on Low	Suction	kpressure	kpressure	l lligh Level	furbine	Steam Flow	Steam Tunnel
·. ·			ECCS Inf	Initiati	ip on Low	I High Bac	n High Bac	on Vesse	re on Low	re on Iligh	re on High
- ·			Low Pressure Nigh Drywe	Low Pressure Water Leve	FW Pumps Tri Pressure	IIPCT Trip or	RCIC Trip of	Turbine Tri	MSIVs Closur Inlet Pres	HSIVs Closu	HSIVs Closu Temperatu

	#13 LBA, ECCS OVERT Brk	×	4	Τ	X	1	X	X	XS	X	s
	#12 S5A, Depress. & ECCS Over., Operator Error		1						X	s	
-	#11 SSA, HPCI, HPCI L8 Trip Failure		1/2		×	s	X		×	s	
*	#10 SBA, HPCS, #20 SBA, HPCS,		MA		X	AN X	NA		×	NA	•
-	#9 SBA, RCIC, E RCIC L& Trip Failure		S		X	XSX	AN X	X	× ×	S	1
	#8 HZT B+K 02C		1					X	S		
	<pre>#7 Alt. Shutdown Cooling, 64000 Suction 96451isvsnU</pre>	:	.								
*	#6 Transient RCIC Hd. Spr.					1		X	NA		
*	#5 Transient HPCS, HPCS L8 Trip Failure		XNA		X	X NA	AN	1	NA		
-	#4 Transient RCIC, RCIC L8 Trip Failure		X			N S	AN X	< s	s	S	
	#3 Transient HPCI, HPCI L8 Trip Failure		X		~	N S	</td <td>S</td> <td>S</td> <td>× s</td> <td></td>	S	S	× s	
	#2 Press. Reg. Fail.				×			-	N S		
	#1 FW Cont. Fail., FW L8 Trip Failure		X		×				×		
	ZERUTAER TNAL9		High Water Level 7 Alarm	High Drywell Pressure Alarm	FW Level 8 Trip .	RCIC Level 8 Trip	HPĊS Level 0 Trip	HPCI Level 8 Trip	HPCI/S and RCIC Initiation on Low Water Level	HPCI/S initiation on High Drywell Pressure	RCIC Initiation on Nigh Drywell Pressure

KEY: X - Feature considered in Base Case Analysis S - Feature in Plant Specific Design NA - Not Applicable

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The values are likely to be extensively cycled in a controlled depressurization mode in a plant-specific application. Was this mode simulated in the test program? What is the effect of this value cycling on value performance and probability of the value to fail open or to fail closed?

### **RESPONSE TO NRC QUESTION 5**

The BWR safety/relief valve (SRV) operability test program was designed to simulate the alternate shutdown cooling mode, which is the only expected liquid discharge event for E. I. Hatch. The sequence of events leading to the alternate shutdown cooling mode is given below.

Following normal reactor shutdown, the reactor operator depressurizes the reactor vessel by opening the turbine bypass valves and removing heat through the main condenser. If the main condenser is unavailable, the operator could depressurize the reactor vessel by using the SRV's to discharge steam to the suppression pool. If SRV operation is required, the operator cycles the valves in order to assure that the cooldown rate is maintained within the technical specification limit of 100°F per hour. When the vessel is depressurized, the operator initiates normal shutdown cooling by use of the RHR system. If that system is unavailable because the valve on the RHR shutdown cooling suction line fails to open, the operator initiates the alternate shutdown cooling mode.

For alternate shutdown cooling, the operator opens one SRV and initiates either an RHR or core spray pump utilizing the suppression pool as the suction source. The reactor vessel is filled such that water is allowed to flow into the main steam lines and out of the SRV and back to the suppression pool. Cooling of the system is provided by use of an RHR heat exchanger. As a result, an alternate cooling mode is maintained.

In order to assure continuous long term heat removal, the SRV is kept open and no cycling of the valve is performed. In order to control the reactor vessel cooldown rate, the operator is instructed to control the flow rate into the vessel. Consequently, no cycling of the SRV is required for the alternate shutdown cooling mode, and no cycling of the SRV was performed for the generic BWR SRV operability test program.

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The ability of the E. I. Hatch SRV to be extensively cycled for steam discharge conditions has been confirmed during steam discharge qualification testing of the valve by the valve vendor. Based on the qualification testing of the SRV's, the cycling of the valves in a controlled depressurization mode for steam discharge conditions will not adversely affect valve performance and the probability of the valve to fail open or closed is extremely low.

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Describe how the values of value  $C_v$ 's in report NEDE-24988-P will be used at E. I. Hatch. Show that the methodology used in the test program to determine the value  $C_v$  will be consistent with the application at E. I Hatch.

# RESPONSE TO NRC QUESTION 6

The flow coefficient,  $C_v$ , for the Target Rock Model 7567F safety relief valve (SRV) utilized in E. I. Hatch was determined in the generic SRV test program (NEDE-24988-P). The average flow coefficient calculated from the test results for the Target Rock Model 7567F, is reported in Table 5.2-1 of NEDE-24988-P. This test value has been used by Georgia Power to confirm that the liquid discharge flow capacity of the E. I. Hatch SRV's will be sufficient to remove core decay heat when injecting into the reactor pressure vessel (RPV) in the alternate shutdown cooling mode. The C<sub>v</sub> value determined in the SRV test demonstrates that the E. I. Hatch SRV's are capable of returning the flow injected by the RHR or CS pump to the suppression pool.

If it were necessary for the operator to place the E. I. Hatch plant in the alternate shutdown cooling mode, he would assure that adequate core cooling was being provided by monitoring the following parameters: RHR or CS flow rate, reactor vessel pressure and reactor vessel temperature.

The flow coefficient for the Target Rock Model 7567F valve reported in NEDE-24988-P was determined from the SRV flow rate when the valve inlet was pressurized to approximately 250 psig. The valve flow rate was measured with the supply line flow venturi upstream of the steam chest. The  $C_v$  for the valve was calculated using the nominal measured pressure differential between the valve inlet (steam chest) and 3' downstream of the valve and the corresonding measured flowrate. Furthermore, the test conditions and test configuration were representative of E. I. Hatch plant conditions for the alternate shutdown cooling mode, e.g. pressure upstream of the valve, fluid temperature, friction losses and liquid flowrate. Therefore, the reported  $C_v$  values are appropriate for application to the E. I. Hatch plant.

# REFERENCES

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- F. L. Leverenz, "Probabilistic Evaluation of High Pressure Liquid Challenge of Safety/Relief Valve Piping", Science Applications, Inc. Palo Alto, California, April 1981.
- Letter to D. G. Eisenhut (USNRC) from T. D. Keenan (BWR Owners Group), November 14, 1979.
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