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# German Standard Problem 4a



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Prepared by W. B. Walton, G. H. Howard, B. Jonsson

ANCO Engineers, Incorporated

Prepared for  
U.S. Nuclear Regulatory  
Commission

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## ABSTRACT

This study deals with structural piping response to dynamic loads generated by hydraulic transients in nuclear power plants. Transients were induced by means of a rupture disc and closure of a feedwater check valve at the Heissdampfreaktor, West Germany. Blind predictions of piping response were made using the computer code EASE2. Comparisons between computer simulations and experimental observations are contained in the report.



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## PREFACE

The following study was conducted by ANCO Engineers, Inc. for the Division of Reactor Safety Research (Dr. John O'Brien, Project Officer), Nuclear Regulatory Commission as part of a joint research effort supported by the NRC and the Ministry of the Interior (BMI) of the Federal Republic of Germany. This report is one of a series of studies conducted by ANCO Engineers as part of the Heissdampfreaktor safety research program. Primary responsibility at ANCO for the work reported herein rests with Mr. William B. Walton, Dr. George H. Howard, and Mr. Blake Johnson. The authors gratefully acknowledge the support provided by Dr. W. Winkler and Dr. T. Grillenberger of Gesellschaft für Reaktorsicherheit, Garshing, as well as Dr. L. Malcher and Dr. Katzenmeier of Kernforschungszentrum Karlsruhe.

## 1.0 INTRODUCTION

At the request of the Ministry of the Interior (BMI) Federal Republic of Germany, the Society for Reactor Safety (GRS) formulated suggestions dealing with, among other things, the formulation and execution of a Standard Problem in the area of pressure wave propagation in nuclear power plant piping systems. Ultimately, German Standard Problem No. 4 (DSP4) was defined and funded. The aim of this problem was to study the closing behavior of a feedwater check valve and the fluid dynamics in its respective pipeline as a result of a simulated break in the pipeline and ensuing rapid closure of the check valve. The tests were to be performed at the Heissdampfreaktor (HDR) using the URL pipe system.

In connection with this study an additional German Standard Problem (4a) was defined. This study was to be a part of the same tests performed for problem 4. The emphasis here would be the determination of the structural dynamic behavior of the pipe system (measurement and calculation of pipe displacements, accelerations, strains/stresses) when subjected to fluid forces from the problem 4 event.

As a part of German Standard Problem 4a (DSP4a), ten (10) engineering firms performed "blind" calculations to predict the structural dynamic response of the URL piping system to problem 4 fluid forces. The results of these calculations were sent via magnetic tape to GRS prior to distribution by GRS of the embargoed experimental results. Chapter 6 of this report presents a very brief comparison of ANCO's predictions to the subsequently released data. ANCO Engineers, Inc. was one of those firms and the sole American participant. This report documents the tasks performed in predicting the URL response.

## 2.0 DESCRIPTION OF GERMAN STANDARD PROBLEM 4a (DSP4a)

DSP4 was an experimental and theoretical simulation of an assumed break in a feedwater line for a nuclear power plant. The simulation was performed using a leg (portion) of the modified recirculating loop of the Heissdampfreaktor (HDR) at Kahl, West Germany (see Figure 2.1). The pipe system was modified by rebuilding a portion of the external forced circulation loop. This involved installing circulation pump Z 199 (Item 7 in Figure 2.1).

The test involved circulating compressed water\* through a portion of the pipe system. The circulation started from the S-support 3 (see Figure 2.1) and travelled to a spherical piece 12, and on through locations 6, 7, 5, 11, 10, 4, 9, and back to the reactor vessel 2. Once steady-state fluid conditions were achieved, the break in the feedwater line was simulated. This involved: (1) circulation pump shut off; (2) fracture of the rupture disc (8 in Figure 2.1, simulated assumed break); (3) closing of rapid shutoff valve 5; and (4) eventual closure of the check-valve 4. At check-valve closure, pressure waves were generated which propagated through the system.

The pressure waves generated during the test were of such amplitude that a high stress state in the pipe material resulted (high relative to the yield stress of the material). For this reason it was of importance to understand the structural dynamic behavior of the pipe system during the fluid dynamic event. This is where DSP4a ties in with DSP4. For DSP4a, the structural dynamic behavior of the URL test pipe line was to be theoretically determined; applied structural loading was to be determined from the

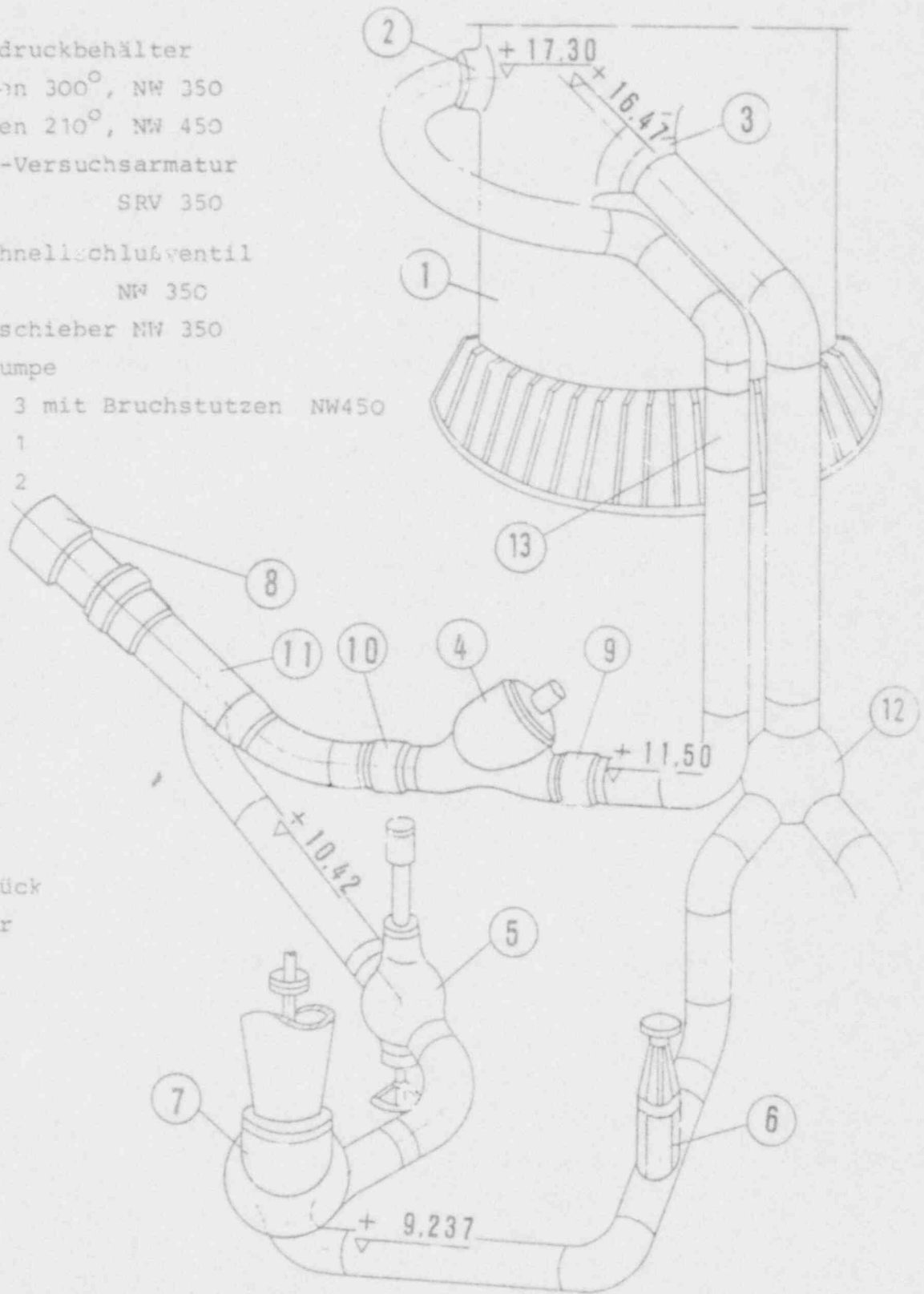
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\*Note: The state of the water in the test line during the circulation phase was 70 bar (10<sup>1.5</sup> psi), 220°C (428°F), and 4 m/sec (13.1 ft/sec; in opposite direction to blowdown flow direction).

FIGURE 2.1: MODIFIED PRIMARY COOLANT LOOP AT HDR

Legende

- ① Reaktordruckbehälter
- ② T-Stutzen 300°, NW 350
- ③ S-Stutzen 210°, NW 450
- ④ Sompell-Versuchsarmatur  
SRV 350
- ⑤ Allo-Schnellschlußventil  
NW 350
- ⑥ Absperrschieber NW 350
- ⑦ Umwälzpumpe
- ⑧ Meßring 3 mit Bruchstutzen NW450
- ⑨ Meßring 1
- ⑩ Meßring 2
- ⑪ T-Stück
- ⑫ Kugelstück
- ⑬ Testrohr



DSP4 test data.\* During the blowdown test the URL pipe was instrumented with displacement transducers, accelerometers, and strain gauges. The data from these instruments is the basis against which all DSP4a theoretical simulation results are to be compared.

The portion of the URL system involved in the structural simulation is shown in Figure 2.2. The pipe has a total length of 18.80 meters (61.63 ft). The pipe begins at the reactor vessel and travels to a rupture disk at its other end. There is a fixed point near the rupture disk. There are no structural supports (snubbers or struts) attached to the pipe. The dimensions and material specification of the various sections of the pipe are given in Table 2.1. The inner pipe diameter and wall thickness vary from 0.351 to 0.453 meter (1.152 to 1.486 ft) and 0.014 to 0.142 meter (0.046 to 0.466 ft), respectively.

As a part of the theoretical simulation for DSP4a, the specified response quantities are to be calculated; they are displacement, acceleration, and stress. The components of stress of interest are the bending, tensile, torsional, tangential, and comparative (von Mises) stresses. Also, the bending axis angle was desired. Figure 2.3 and Table 2.2 describe the desired quantities.

The fluid forces on the pipe, due to the pressure waves, were to be calculated using the DSP4 test data (i.e., fluid pressure). The pressure as a function of time is known at numerous locations, particularly at inlets and/or outlets of pipe elbows. The method used for obtaining the forces was to be developed by each investigator. Thus, DSP4a involves determining both the fluid/structure interface forces and the structural response.

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\*Note: The calculations performed by ANCO Engineers, Inc. were to simulate reality as closely as possible (they were not designed to be conservative). Also, no previous (prior to completion of the analysis) knowledge of the structural dynamic test results was known.





TABLE 2.1

## DIMENSIONS AND MATERIAL DESIGNATIONS FOR PIPE SECTIONS

Lfd.* Nr.	Teilstück	Werkstoff	Nennweite	Innendurch- messer [ mm ]	mittlere x) Wanddicke [ mm ]	Teilstück- länge [ mm ]
1	RDB-Wand	23NiMoCr36	-	-	142	142
2	T-Stutzen	23NiMoCr36	NW 350	351,2	104,4/27,6	375
3	120°-Bogen	Nr. 4550	NW 350	360	25,0	1120
4	Rohrstück mit 30°-Biegung	Nr. 4550	NW 350	360	19,3	2508
5	60°-Bogen	Nr. 4550	NW 350	360	25,0	560
6	Rohrstück	Nr. 4550	NW 350	360	19,3	1494
7	90°-Bogen	Nr. 4550	NW 350	360	25,0	840
8	Rohrstück	Nr. 4550	NW 350	360	19,3	630
9	Probenstück	WB 35	NW 350	377	14,0	1000
10	Rohrstück	Nr. 4550	NW 350	360	19,3	2925
11	Übergangsstück	15Mo3	NW 350	360	19,3/25,0	100
12	90°-Bogen	15Mo3	NW 350	356,4	25,0	957
13	Rohrstück	15Mo3	NW 350	371,4	17,5	510
14	Meßring I	15Mo3	-	371,4	-	480
15	SRV 350	GS C25	NW 350	-	-	1250
16	Meßring II	15Mo3	-	371,4	-	500
17	Rohrstück	15Mo3	NW 350	371,4	17,5	260
18	61°-Bogen	15Mo3	NW 350	356,4	25,0	648
19	T-Stück	15Mo3	NW450/350	428/371,4	40,0/17,5	1300
20	Stutzen	15Mo3	NW 450	453	61,0	520
21	Meßring III m. Bruchstutzen	15Mo3	NW 450	453	27,5/93,5	740
22	Berstscheiben u. Mündungsrohr		NW 450	453	-	250
						19109

x) Die mittlere Wanddicke wurde aus den Angaben der Rohrleitungszeichnungen entnommen und weicht erheblich von den an einzelnen Stellen durchgeführten MPA-Messungen ab (siehe Anlage 12).

Die beim Probenstück (lfd. Nr. 9) angegebenen Maße sind Istmaße.

\* Note: These sections are defined in the next page of this table.

TABLE 2.1 (Continued)

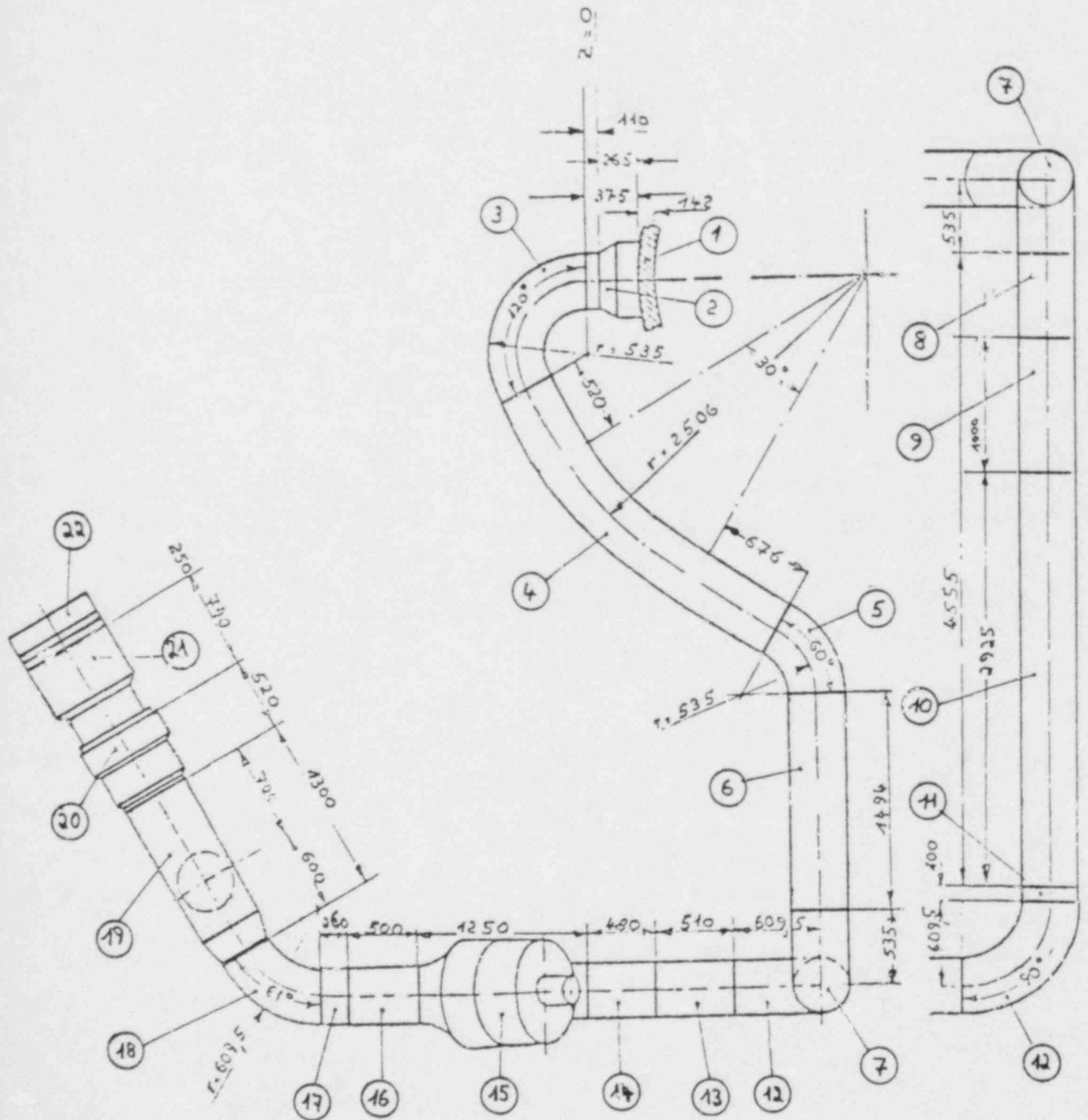


FIGURE 2.3: Desired Structural Response Quantities.

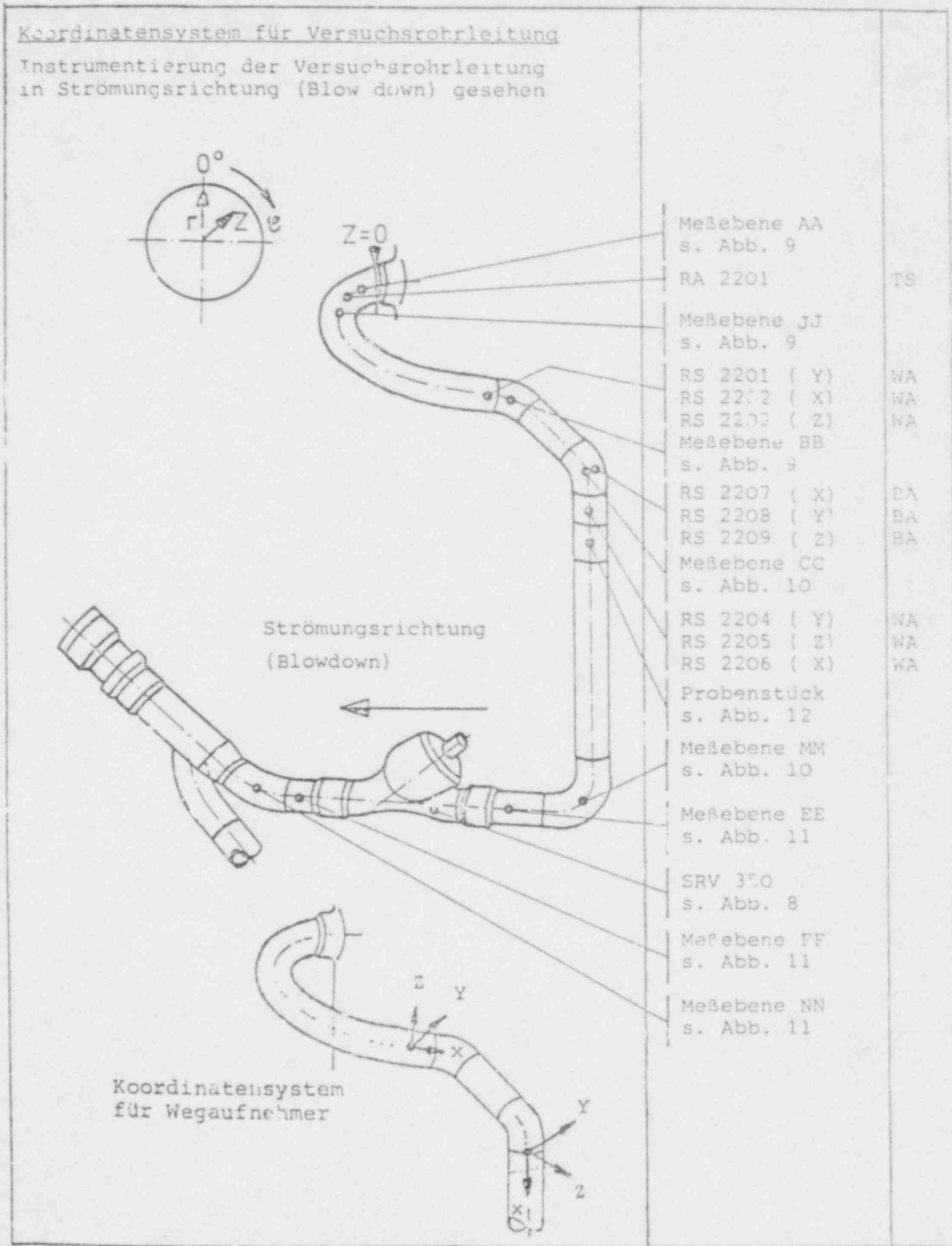


TABLE 2.2: DESIRED STRUCTURAL RESPONSE QUANTITIES

Entsprechende Meßstelle	zu ermittelnde Größe	
RS 2201	Weg in y-Richtung	} (3,4 m vom RDB-Stützen)
RS 2202	Weg in x-Richtung	
RS 2203	Weg in z-Richtung	
RS 2204	Weg in y-Richtung	} (7,7 m vom RDB-Stützen)
RS 2205	Weg in z-Richtung	
RS 2206	Weg in x-Richtung	
SS 4004	Weg in z-Richtung	} am Ventil
SS 4005	Weg in x-Richtung	
SS 4006	Weg in y-Richtung	
SS 4001	Beschleunigung in x-Richtung	} am Ventil
SS 4002	Beschleunigung in y-Richtung	
SS 4003	Beschleunigung in z-Richtung	
RK 1010	Zugspannung	} Meßebene A
RK 2010	Biegespannung	
RK 2210	Winkel der Biegeachse	
RK 3010	Torsionsspannung	
RK 4110	Tangentialspannung durch Innendruck	
RK 5010	Vergleichsspannung	
RK 1011	Zugspannung	} Meßebene C1
RK 2011	Biegespannung	
RK 2211	Winkel der Biegeachse	
RK 3011	Torsionsspannung	
RK 4111	Tangentialspannung durch Innendruck	
RK 5011	Vergleichsspannung	
RK 1012	Zugspannung	} Meßebene M1
RK 2012	Biegespannung	
RK 2212	Winkel der Biegeachse	
RK 3012	Torsionsspannung	
RK 4112	Tangentialspannung durch Innendruck	
RK 5012	Vergleichsspannung	
RK 1013	Zugspannung	} Meßebene E
RK 2013	Biegespannung	
RK 2213	Winkel der Biegeachse	
RK 3013	Torsionsspannung	
RK 4113	Tangentialspannung durch Innendruck	
RK 5013	Vergleichsspannung	
RK 1014	Zugspannung	} Meßebene F
RK 2014	Biegespannung	
RK 2214	Winkel der Biegeachse	
RK 3014	Torsionsspannung	
RK 4114	Tangentialspannung durch Innendruck	
RK 5014	Vergleichsspannung	

### 3.0 STRUCTURAL MODEL OF THE URL PIPE SYSTEM

To simulate the dynamic behavior of the URL pipe structure for DSP4a, a linear finite element model was constructed.\* A plot representing the model is shown in Figure 3.1. The model includes the section of the pipe from the reactor vessel wall to the T-piece. Fixed points (all degrees of freedom deleted) were assumed at each end of the pipe. The model was defined using 30 nodes (end nodes fixed; 168 degrees of freedom) and 27 finite elements (straight and curved pipe elements). Appendix A contains a partial listing of the EASE2 input data deck which defines the finite element model.

Various coordinate systems were defined. The global coordinate system (the reference system for mass and stiffness matrix assembly) is given by X, Y, Z (see Figure 3.1). Local coordinate systems were defined for output of displacement and acceleration simulation results in accordance with DSP4a definition (specified by the Society for Reactor Safety; T. Grillenberger); they are shown in Figure 3.2 and are represented by x, y, z.

A flexibility factor  $k_p$  is used in EASE2 to modify the bending terms in the flexibility matrix for the curved pipe element. The flexibility matrix for the curved pipe element is derived totally from curved beam theory. This overestimates the stiffness (as compared to experimental data). To correct for this, the flexibility factor  $k_p$  is used. It is given by:

$$k_p = (1.65/h) / [1 + (6p/Eh)(R/t)^{4/3}] \geq 1$$

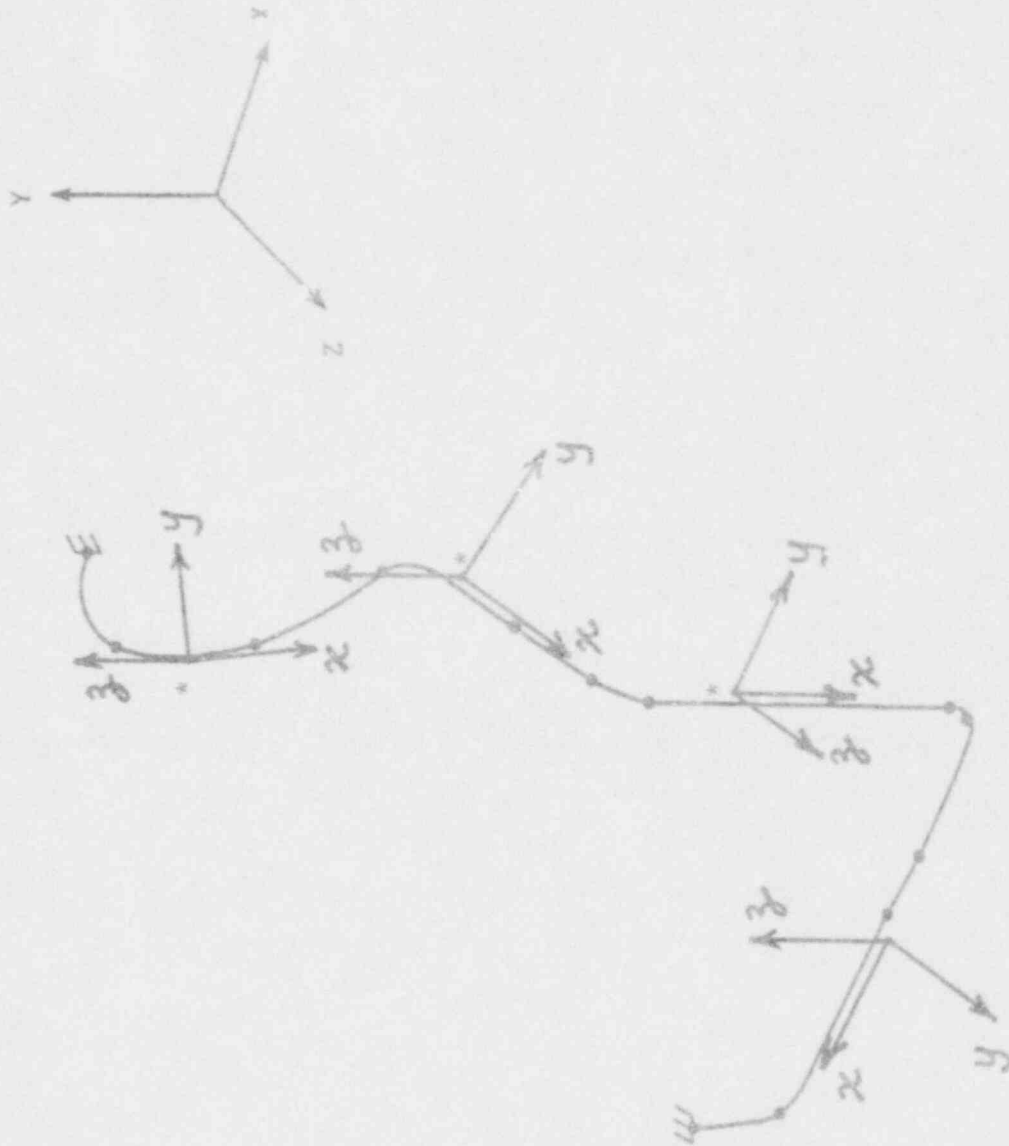
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\*Note: The computer code EASE2 (Elastic Analysis for Structural Engineering) was used. This code is accessible through the CDC Cybernet Network Worldwide.



FIGURE 3.1: ANCO EASE2 Structural Model (Partial List of Node Numbers).

FIGURE 3.2: Local Coordinate Systems.\*



\* Note: These coordinate systems are for output of displacements and accelerations.



where: p = internal pressure  
h =  $tR/r^2$   
r =  $(d_o-t)/2$   
R = bend radius  
t = wall thickness  
 $d_o$  = outside pipe diameter  
E = Young's modulus

This correction can be significant. From experience, errors on the order of 25% have been observed in the first natural frequency of piping systems when this correction was not included.

Following model formulation, an eigenvalue analysis of the URL model was executed. The first ten (10) eigenfrequencies are given in Table 3.1. It is likely that with the number of nodes used in defining the pipe model (30 nodes) that the last few modes are not accurately defined. If it is minimally acceptable to use four points to define a half cycle of a standing wave, then the seventh (7th) mode is approximately the highest mode that can be acceptably defined with the thirty (30) node points. Four points will reasonably well define a half cycle. For three points, it is possible to define up to about the tenth (10th) mode. Three points will marginally define a half cycle. For these reasons, it is reasonable to expect the URL modes to be fairly well defined up to at least the eight (8th) or ninth (9th) mode.

TABLE 3.1

NATURAL FREQUENCIES  
FOR HDR/URL BLOWDOWN MODEL

<u>Mode #</u>	<u>Eigenfrequency (Hz)</u>
1	5.46
2	8.54
3	9.43
4	26.18
5	30.55
6	39.71
7	45.57
8	50.54
9	64.23
10	76.22

#### 4.0 PRESSURE WAVE FORCES ON THE URL PIPE

DSP4a involves accurately simulating the structural dynamic response of the modified URL system when excited by the DSP4 blowdown fluid forces. This type of simulation deals with fluid-structure interaction (the response of a structure due to transient fluid events).

There are two general approaches to the fluid-structure problem. One involves the definition and solution of a coupled problem. For this approach, the equations representing the fluid and structure behavior are coupled to each other. In order to solve for the response of either the fluid or structure, the response of both must be found simultaneously. The other approach involves determining an uncoupled solution. This involves solving for or determining experimentally the fluid response, independently of the structural response. The fluid response is then input into the structural problem, and the uncoupled structural response is determined. The latter of these two approaches was used for DSP4a. This is because the fluid response, determined as a part of DSP4, is to be used as input to DSP4a. The fluid response (i.e., pressure) is to be used to determine the fluid forces on the URL pipe. These forces will be used in determining the structural response.

Given the transient fluid response of a system, there are at least two possible basic approaches for determining the fluid forces (uncoupled fluid-structural problem) on a pipe system. One approach could be called the "equivalent concentrated forces method." Basically it says, given a continuous fluid stress distribution in a pipe (at the pipe fluid interface), it is possible to determine equivalent concentrated loads at the node points used to define the structural model. This is done, in part, by applying the following equation:

$$P_{eq} = \int_s a^T \phi dS$$

where  $P_{eq}$  is the equivalent end load vector for a pipe element,  $a^T$  is the transpose of the matrix  $a$ , where  $a$  gives the relationship, for a discrete structural element (i.e., finite element), between the spatially continuous interior displacements and the discrete element displacements (at the node points),  $\phi$  is the matrix of distributed surface stresses (distributed field of fluid stress), and  $dS$  refers to the integration being carried out over the applicable surface ( $s$ ). This expression is only for a single element. A  $P_{eq}$  would have to be determined for each element. Then, a global  $P_{eq}$  would need to be determined for the entire structure. This is an excellent approach, provided the distributed loading can be determined for the entire structure. This may be possible for the DSP4a, but would be very difficult, if at all possible. As this is a wave propagation problem, it greatly complicates the determination of a continuous pressure distribution. Because of this, it was decided to use a second approach to solving the uncoupled problem.

Another of the basic uncoupled fluid-structure approaches involves the use of the linear momentum equation from fluid mechanics (control volume formulation):

$$\bar{F}_s + \iiint_{C.V.} \bar{B}(\rho dv) = \iint_{C.S.} \bar{V}(\rho \bar{V} \cdot d\bar{A}) + \frac{\partial}{\partial t} \iiint_{C.V.} \bar{V}(\rho dv)$$

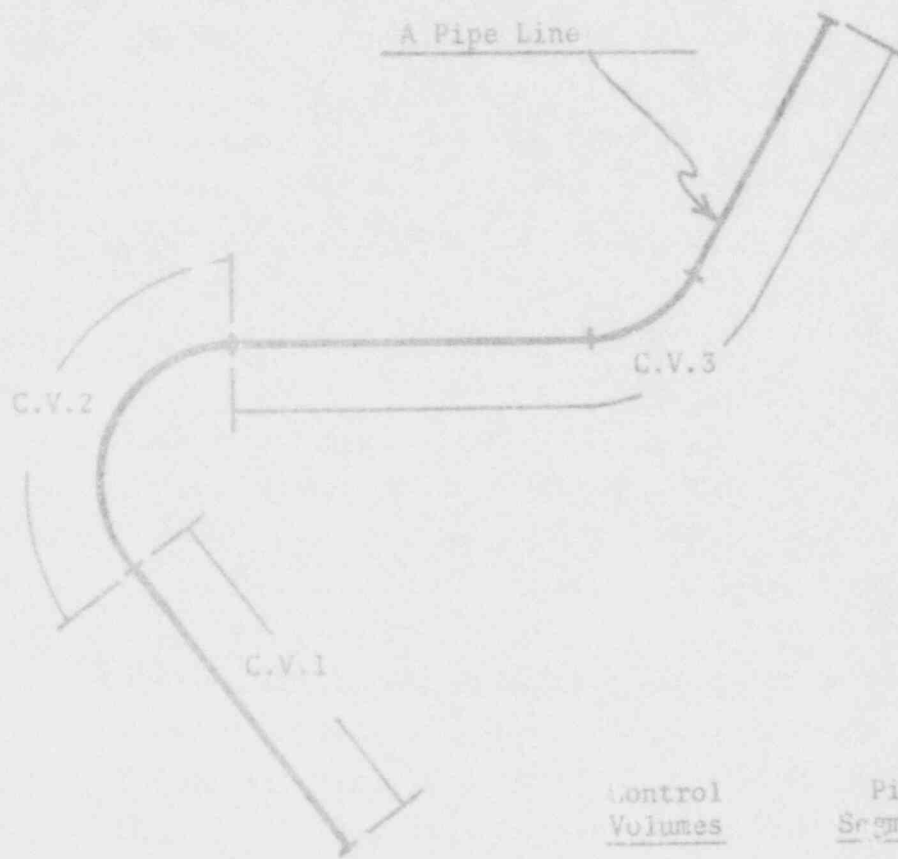
where  $\bar{F}_s$  is the total surface force on the control surface (C.S.),  $\bar{B}$  is the body force distribution,  $\bar{V}$  is the fluid velocity,  $d\bar{A}$  is a differential area normal to the C.S. and  $\rho$  is the fluid density.

To apply this equation, the space occupied by the fluid in the pipe was subdivided into control volumes (the union of the control volumes is equal to the total space in the pipe). There are an infinite number of ways to subdivide the space. Various methods for doing this have been devised.

The control volumes for this problem obviously had to be contained in straight, curved, and combinations of straight and curved lengths of pipe (see Figure 4.1). One constraint on the selection of volumes was that the pressure had to be known at each of its two ends. (As will be discussed later, this was not necessary for straight volumes.) This is necessary for evaluating the surface force  $\bar{F}_s$ .

The question arises as to how large the control volumes can be (or to how small they should be). For each control volume, there is essentially one resultant force. If there are only a few control volumes, there will only be a few forces representing the spatially continuous load distribution on the pipe. The smaller the control volumes the more closely their combined loading resembles the continuous loading.

In choosing the size of a control volume it is important to consider the degree to which the center of pressure of its respective fluid changes location during the fluid dynamic event. The control volume resultant force should be applied at the center of pressure. This can be done by having a fine mesh for the discrete structural model, especially for such elements as curved pipes (locations where some of the largest resultant forces are generated). During the structural simulation the resultant force can be moved from node to node. Even though these calculations can be performed with enough data, they are very involved (i.e., the center of pressure must be calculated at each solution step). This, coupled together with the fact that only a relatively limited amount of DSP4 data is available, the resultant control volume force, for DSP4a, was not applied at the center of pressure, but approximately at the geometric center of the control volume. If the control volumes are chosen to be too large there will possibly be considerable error in the force application point locations.



<u>Control Volumes</u>	<u>Pipe Segments</u>
C.V.1	Straight
C.V.2	Curved
C.V.3	Straight/Curved

FIGURE 4.1: Some Possible Combinations of Pipe Segments Used in Defining Control Volumes.

On the basis of the above discussion, the URL blowdown model (DSP4a) was broken up into six (6) control volumes (see Figure 4.2). As can be seen, four of the volumes are made up of a combination of straight and curved sections. Two of the volumes are straight. The point of application of the resultant force for a given control volume is represented by the designator LP (load point). No load point is defined for volume six (6). This volume is straight with the only forces it can exert on its section of pipe being frictional. For a DSP4 type of event, the frictional fluid forces are negligible.

In determining the control volume forces on the pipe, it is necessary to evaluate the momentum efflux rate, rate of change of control volume momentum, and body force integrals for each control volume. The two momentum integrals are dependent on mass flow rates. The major events that take place during the DSP4 test that might effect the mass flow rate are: (1) the circulation pump gives a constant flow rate of compressed liquid of approximately 1600 m<sup>3</sup>/hour (15.7 ft /sec); (2) the pump is shut off at the beginning of the test, resulting in essentially zero flow rate; (3) fracture of the rupture disc, resulting in flashing of the compressed liquid at the disc; (4) closing of the rapid shut-off valve immediately after rupture disc fracture (this insures isolation of the second section of pipe); and (5) closing of the check-valve SRV350, resulting in zero mass flow rate between the fitting and reactor vessel (after closure). The flashing will give rise to large steam and small to moderate compressed liquid velocities. This will generate some force on the pipe, but is not the major forcing generated during the entire event. The major forcing is generated during check-valve closure. For this phase of the forcing, the compressed liquid flow rate is initially at a small to moderate level. As the event progresses, the flow rate will diminish, eventually, to zero. Figure 4.3 gives a hypothetical example of this. With the portion of the DSP4 event that gives rise to the largest forcing, seeing only from moderate to zero net flow rates, a reasonable assumption for calculating peak forces on the URL pipe may be that the fluid flow terms in the momentum equation can be neglected. Also, it is reasonable to assume that the body force distribution will not materially affect the flow.



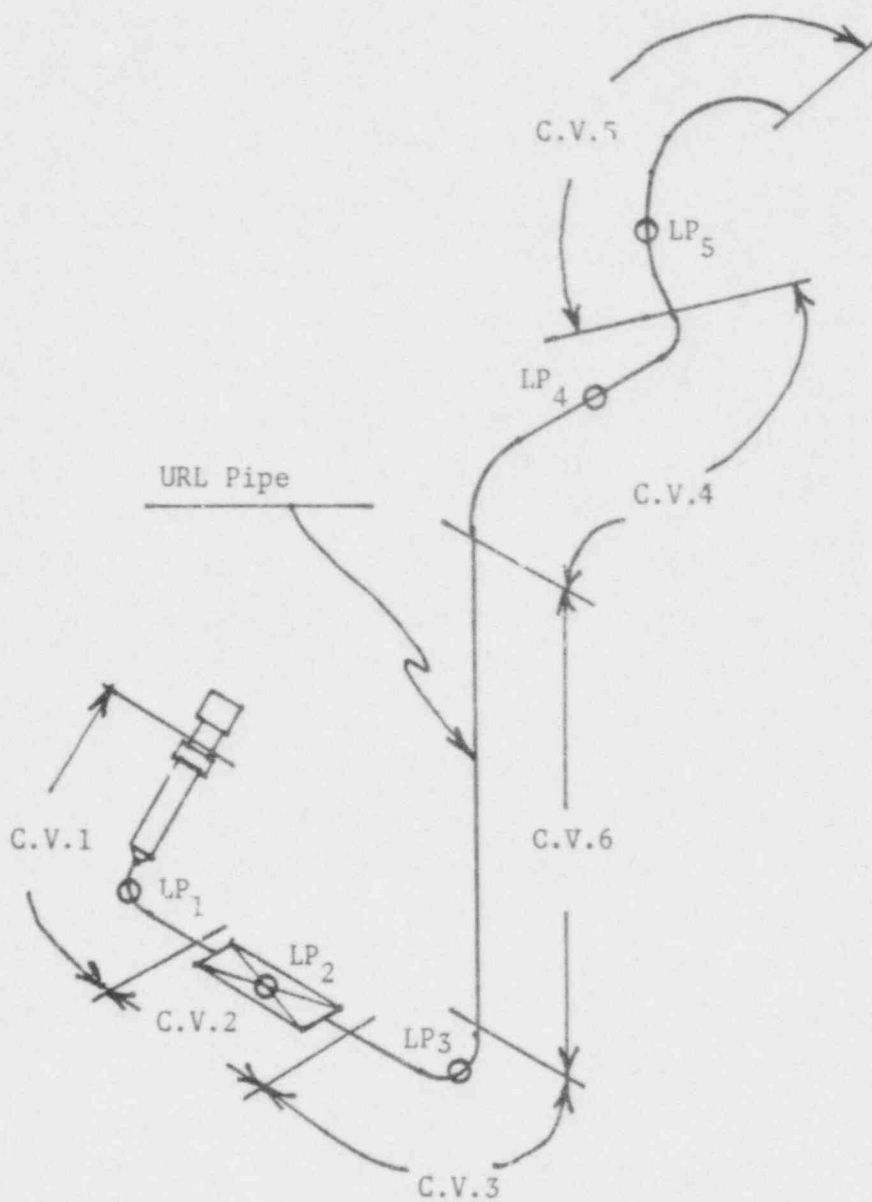
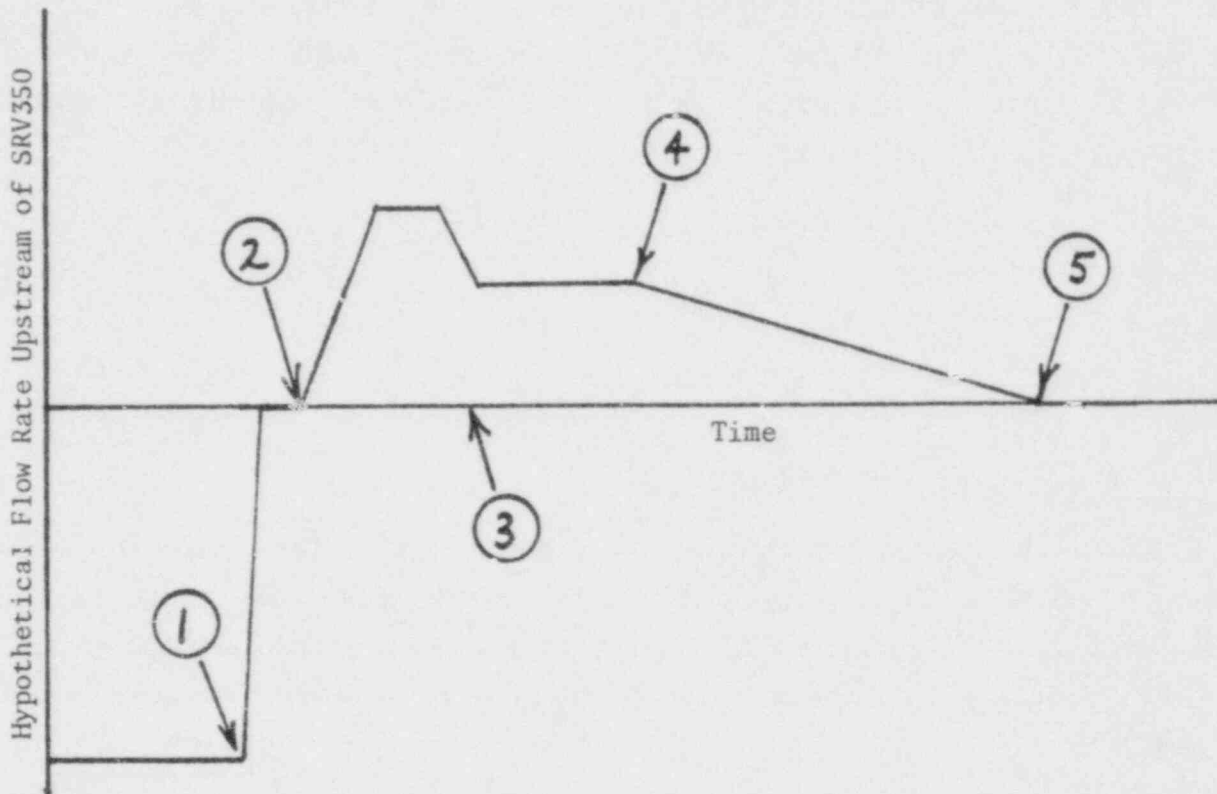


FIGURE 4.2: Control Volumes Used for DSP4a.





<u>Sequence of Events</u>	<u>Description</u>
1	Circulation pump shut off
2	Fracture of bursting disc
3	Closing of rapid shutoff valve
4	SRV350 begins to close
5	Closure of SRV350

FIGURE 4.3: Hypothetical Example of A DSP4 Event.

Using these assumptions, the momentum equation (control volume formulation) reduces to  $\bar{F}_S = 0$  (the sum of the external surface forces equals zero). This equation is applied to an arbitrarily shaped control volume (for a piping system) as shown in Figure 4.4. The only forces of concern are the pressure forces (forces on the control volume from the adjacent fluid) and the force of the pipe on the volume. It is seen that the resultant force of a control volume (using the above assumptions) on its corresponding section of pipe is given by:

$$\bar{F}_R = p_1 A_1 \hat{i}_1 - p_2 A_2 \hat{i}_2$$

This equation is applied to each of the control volumes used to define the URL space. Figure 4.5 defines the pressures and area used in the computations. The equations for the fluid forces (on the pipe) at the load points are given in Table 4.1. Appendix B gives the derivation of these simple equations.

The fluid pressure time history data was used together with the equations in Table 4.1 to calculate the fluid forces on the pipe. The channels of pressure data that were taken to correspond to  $P_1, \dots, P_7$  are listed in Table 4.2. Plots of the calculated fluid forces on the pipe are given in Figures 4.6 through 4.15. Table 4.3 gives the force application point and direction for a given force number (i.e., force number RF0006). Table 4.4 gives the minimum and maximum values of each of the force components.

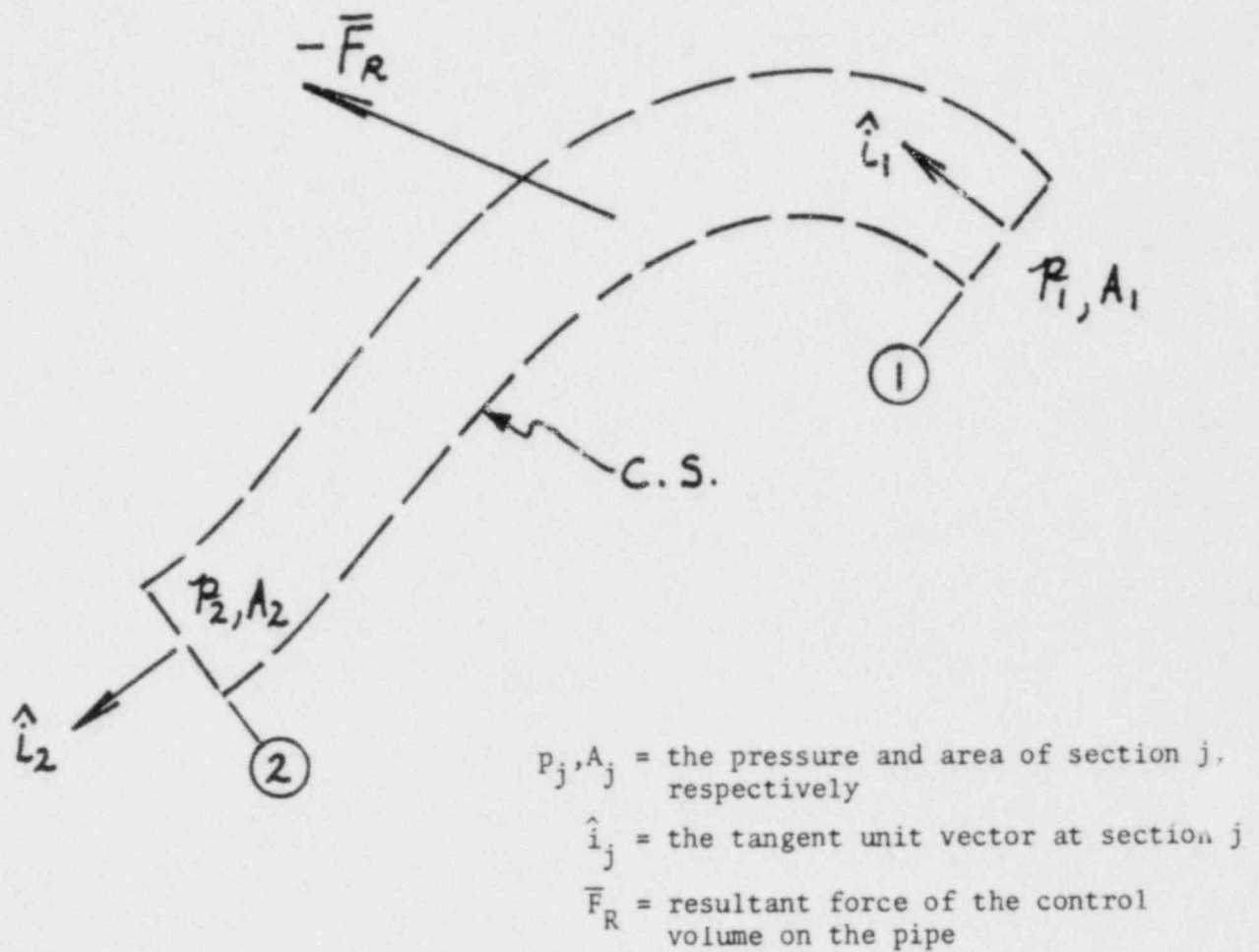


FIGURE 4.4: Arbitrarily Shaped Control Volume for Obtaining Pipe Segment Forces.



TABLE 4.1

## DSP4 FLUID FORCES ON PIPE

<u>Load Point</u>	<u>Fluid Force on Pipe at Load Point*</u>
1	$\bar{F}_1 = (P_2 A_2 - P_1 A_1 \sin 29^\circ) \hat{i} + P_1 A_1 \cos 29^\circ \hat{j}$
2	$\bar{F}_2 = (P_3 A_3 - P_2 A_2) \hat{i}$
3	$\bar{F}_3 = P_5 A_5 \hat{i} + P_4 A_4 \hat{j}$
4	$\bar{F}_4 = P_7 A_7 \cos 60^\circ \hat{i} + P_7 A_7 \sin 60^\circ \hat{j} + P_6 A_6 \hat{k}$
5	$\bar{F}_5 = -P_7 A_7 \cos 15^\circ \hat{i} - P_7 A_7 \sin 15^\circ \hat{j}$

---

\*Note: The unit vectors  $\hat{i}$ ,  $\hat{j}$ ,  $\hat{k}$  correspond to the local coordinate systems at the load points (see Figure 4.6).

TABLE 4.2: RELATION OF PRESSURE  $P_i$  TO TRANSDUCER CHANNEL

<u>Pressure</u>	<u>Corresponding Transducer</u>	<u>Flow Area (m<sup>2</sup>)</u>
P <sub>1</sub>	RP2114	0.1083
P <sub>2</sub>	RP2108	0.1083
P <sub>3</sub>	RP2205	0.1083
P <sub>4</sub>	RP2205	0.1083
P <sub>5</sub>	RP2203	0.1018
P <sub>6</sub>	RP2202	0.1018
P <sub>7</sub>	RP2201	0.1018

FIGURE 4.6

V60.4

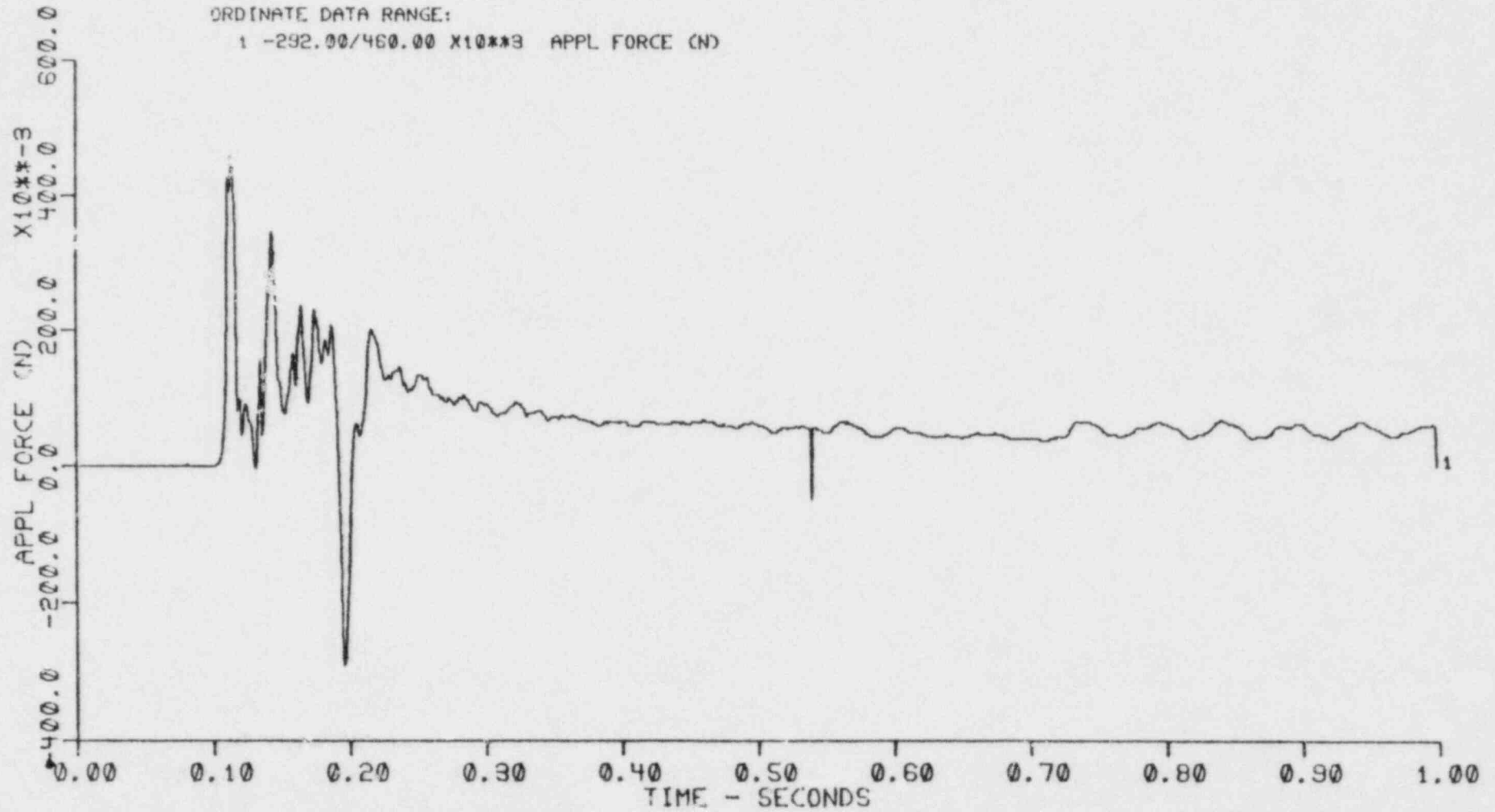
SP4AHDRSRV350

TEST:

RUN:

ORDINATE DATA RANGE:

1 -292.00/460.00 X10\*\*3 APPL FORCE (N)



4-13

CHANNEL 1 RF0001 N

FIGURE 4.7

V60.4

SP4AHDRSRV350

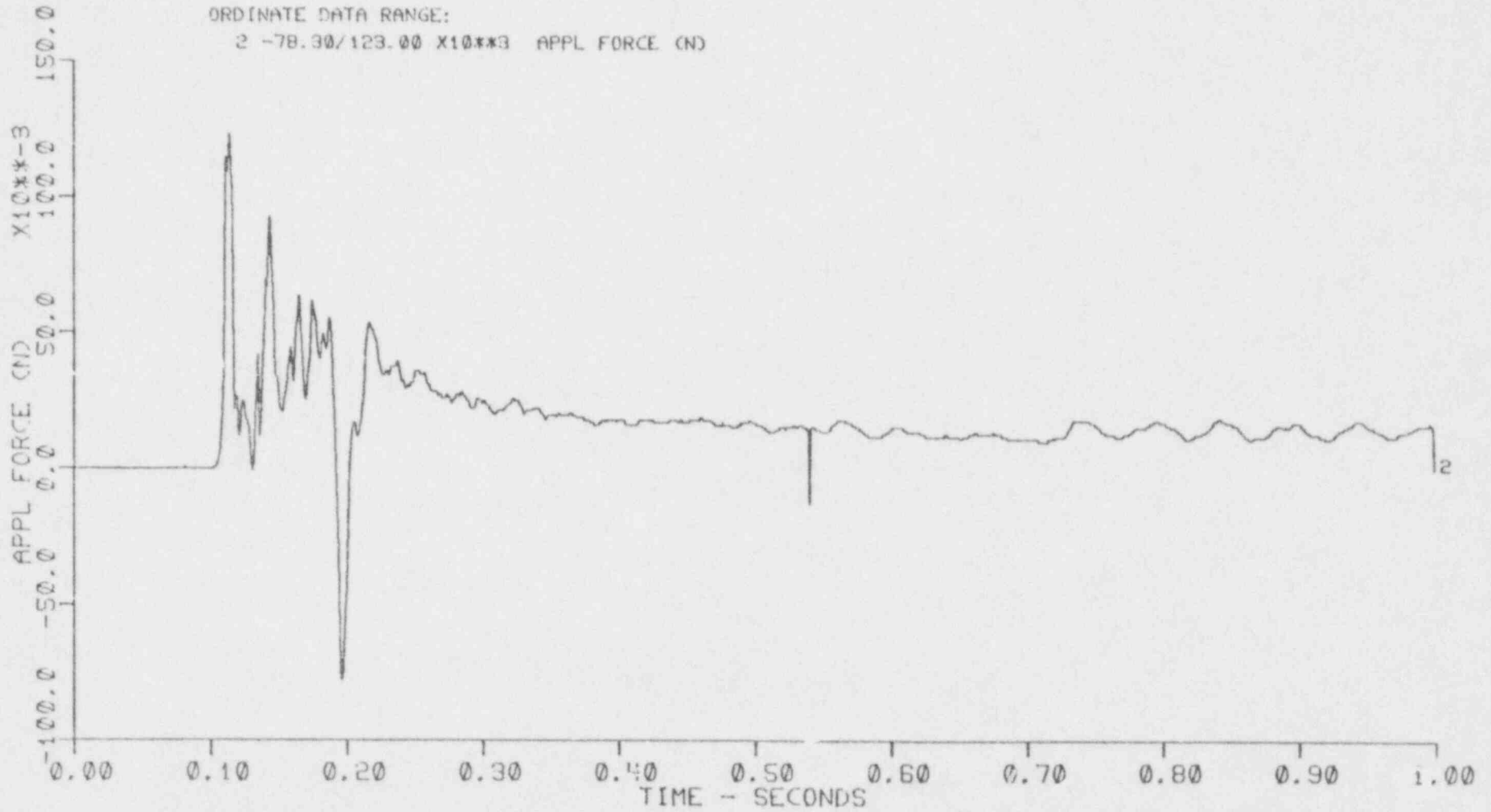
TEST:

RUN:

ORDINATE DATA RANGE:

2 -78.30/123.00 X10\*\*3 APPL FORCE (N)

4-14



CHANNEL 2 RF0002 N



FIGURE 4.8

V60.4

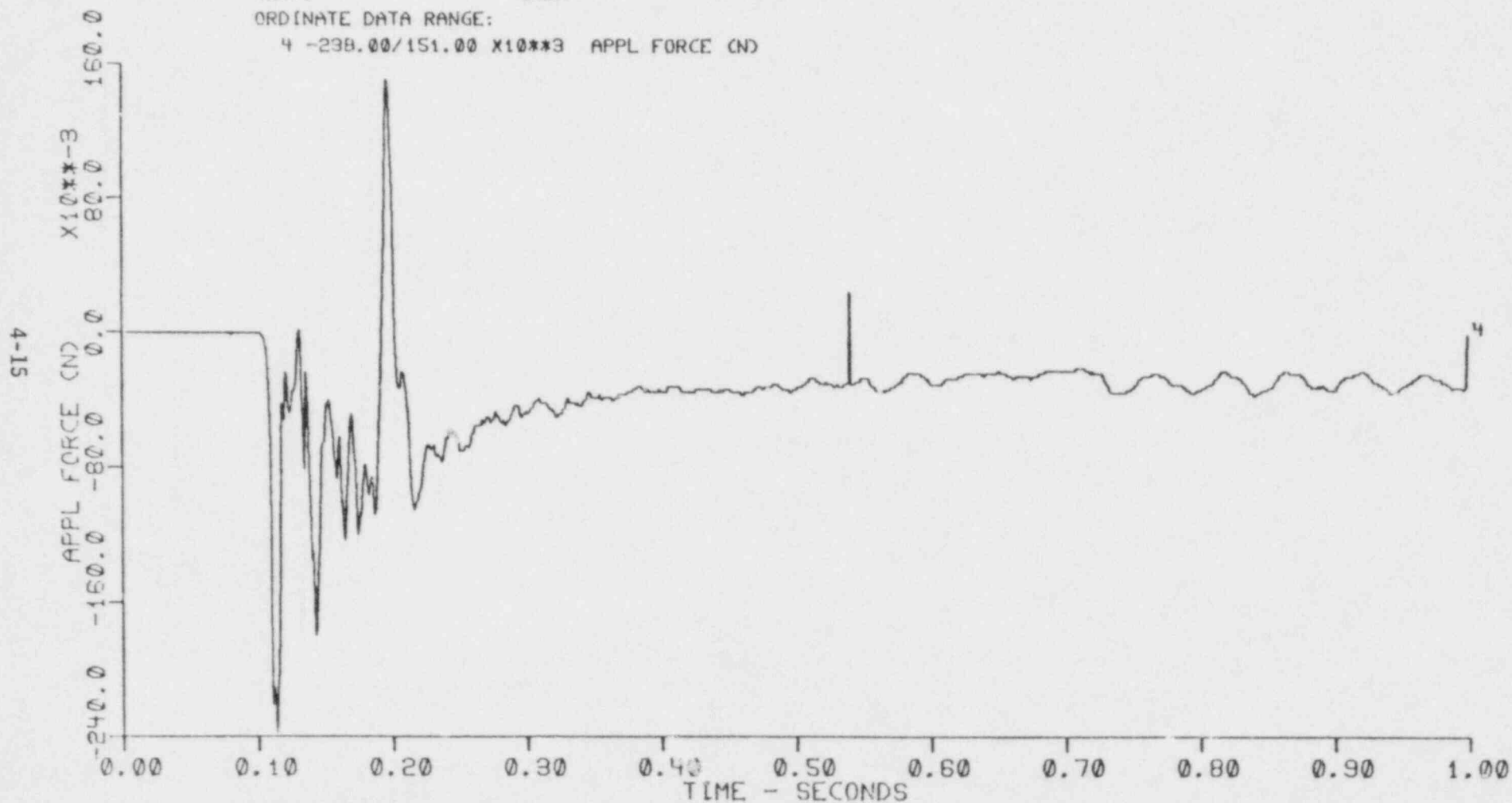
SP4AHDRSRV350

TEST:

RUN:

ORDINATE DATA RANGE:

4 -238.00/151.00 X10\*\*3 APPL FORCE (N)



CHANNEL 4

RF0004

N

FIGURE 4.9

V60.4

SP4AHDRSRV350

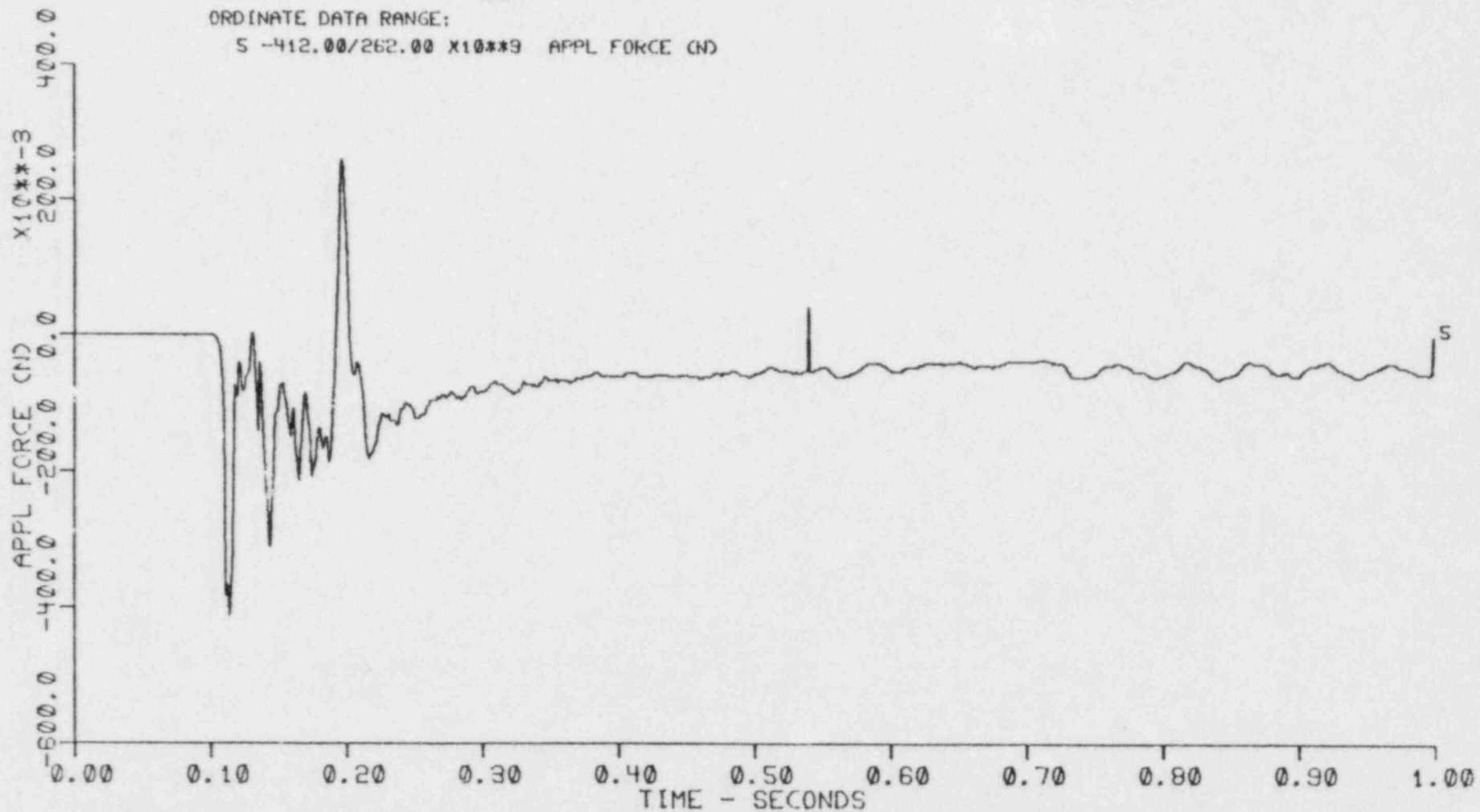
TEST:

RUN:

ORDINATE DATA RANGE:

S -412.00/262.00 X10\*\*9 APPL FORCE (N)

4-16



CHANNEL S RF0005 N

FIGURE 4.10

V60.4

SP4AHDRSRV350

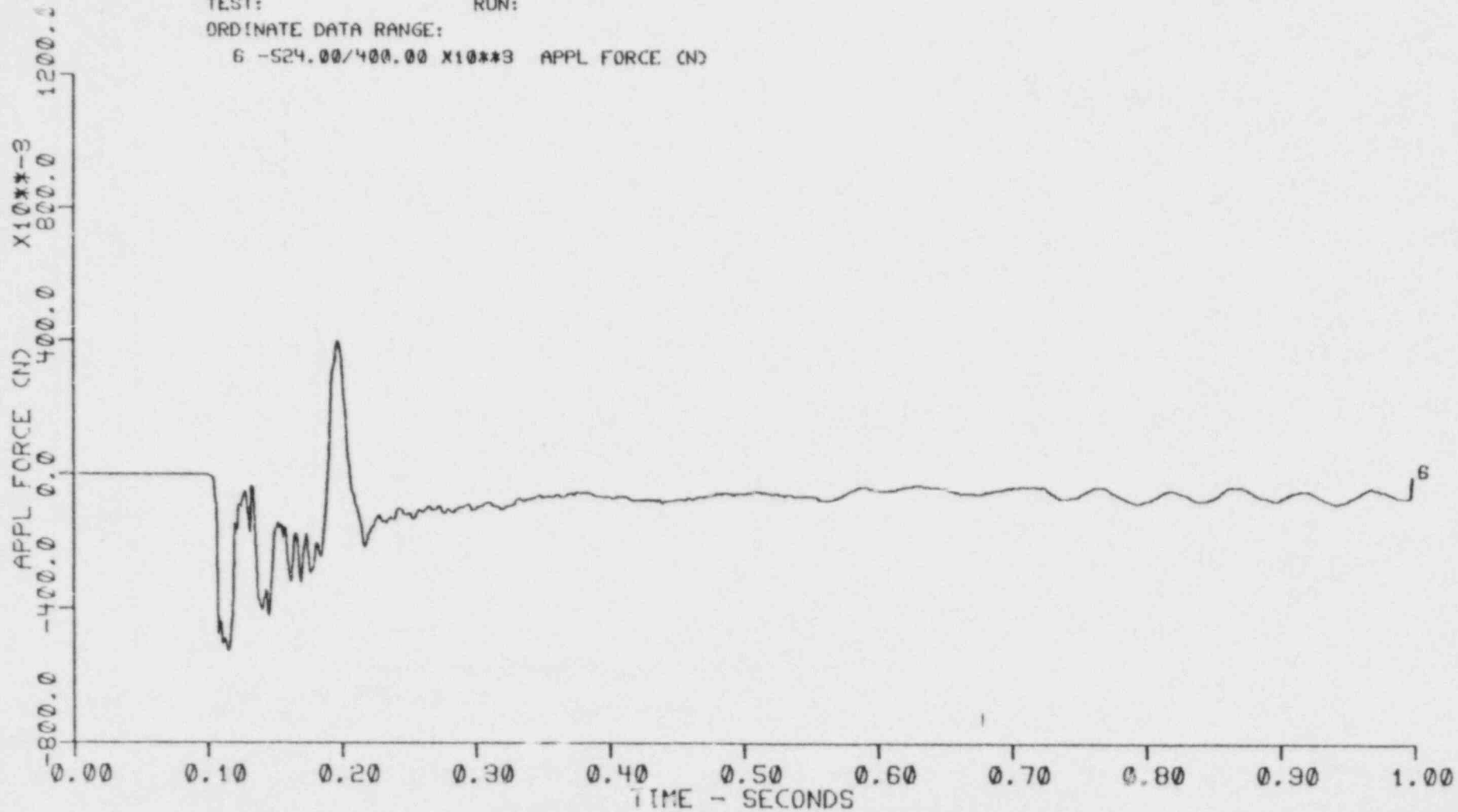
TEST:

RUN:

ORDINATE DATA RANGE:

6 -524.00/400.00 X10\*\*3 APPL FORCE (N)

4-17



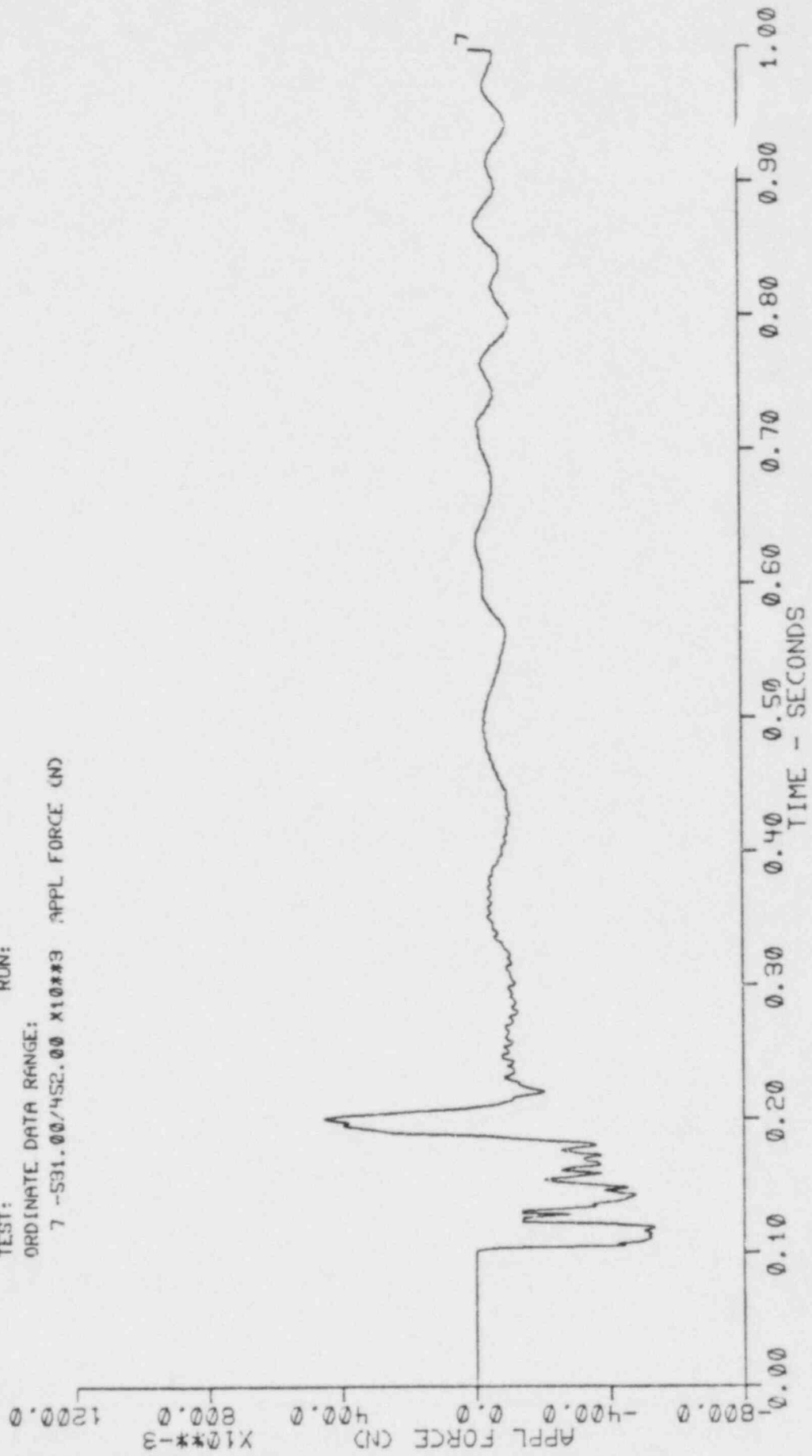
CHANNEL 6

RF0006

N

FIGURE 4.11

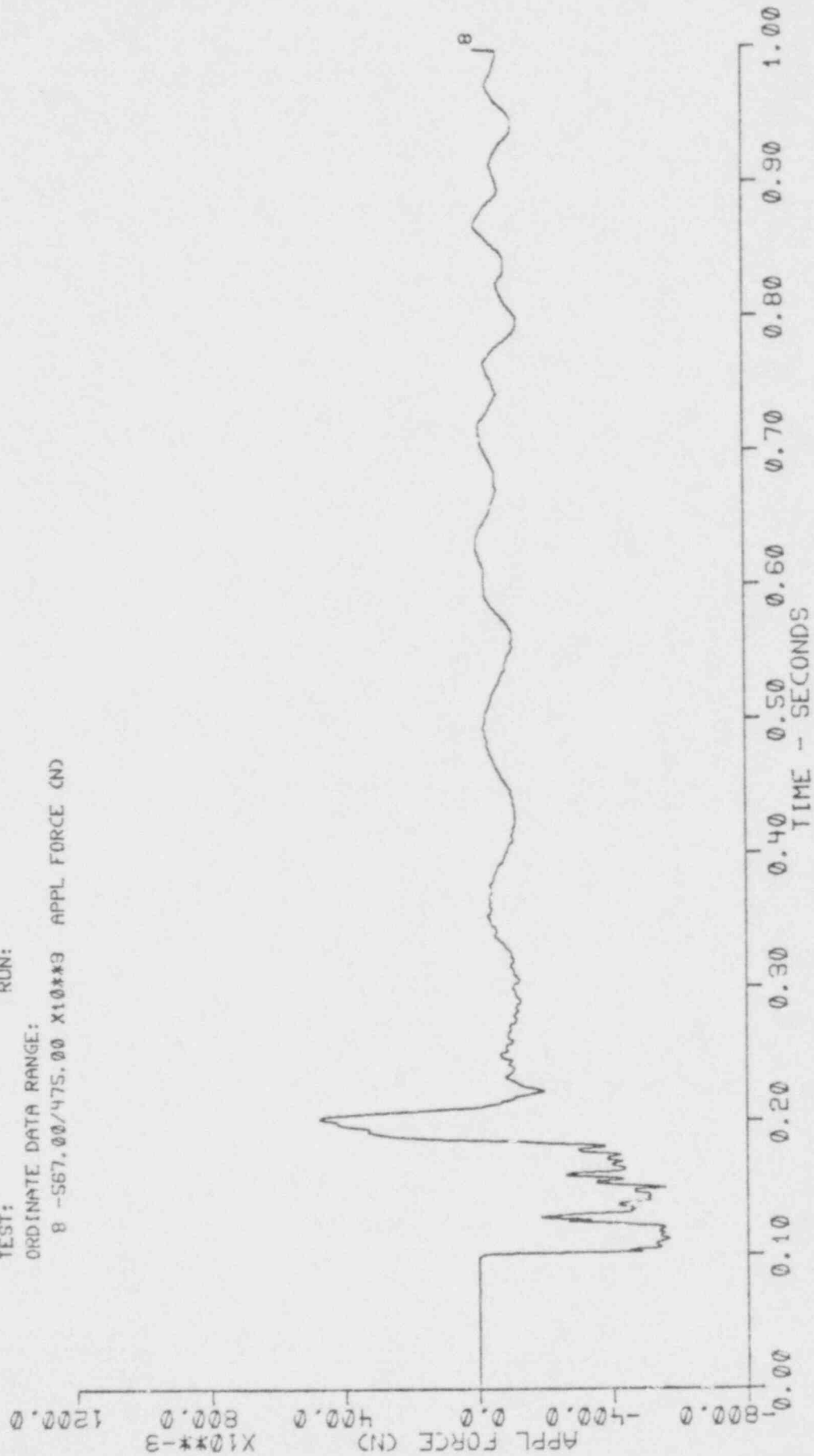
V60 4 SP4AHDRSRV350  
TEST: RUN:  
ORDINATE DATA RANGE:  
7 -531.00/452.00 X10\*\*3 APPL FORCE (N)



CHANNEL 7 RF0007 N

FIGURE 4.12

V60.4  
SP4AHDRSRV350  
RUN:  
TEST:  
ORDINATE DATA RANGE:  
8 -567.00/475.00 X10\*\*9 APPL FORCE (N)



CHANNEL 8 RF008 N

FIGURE 4.13

V60.4

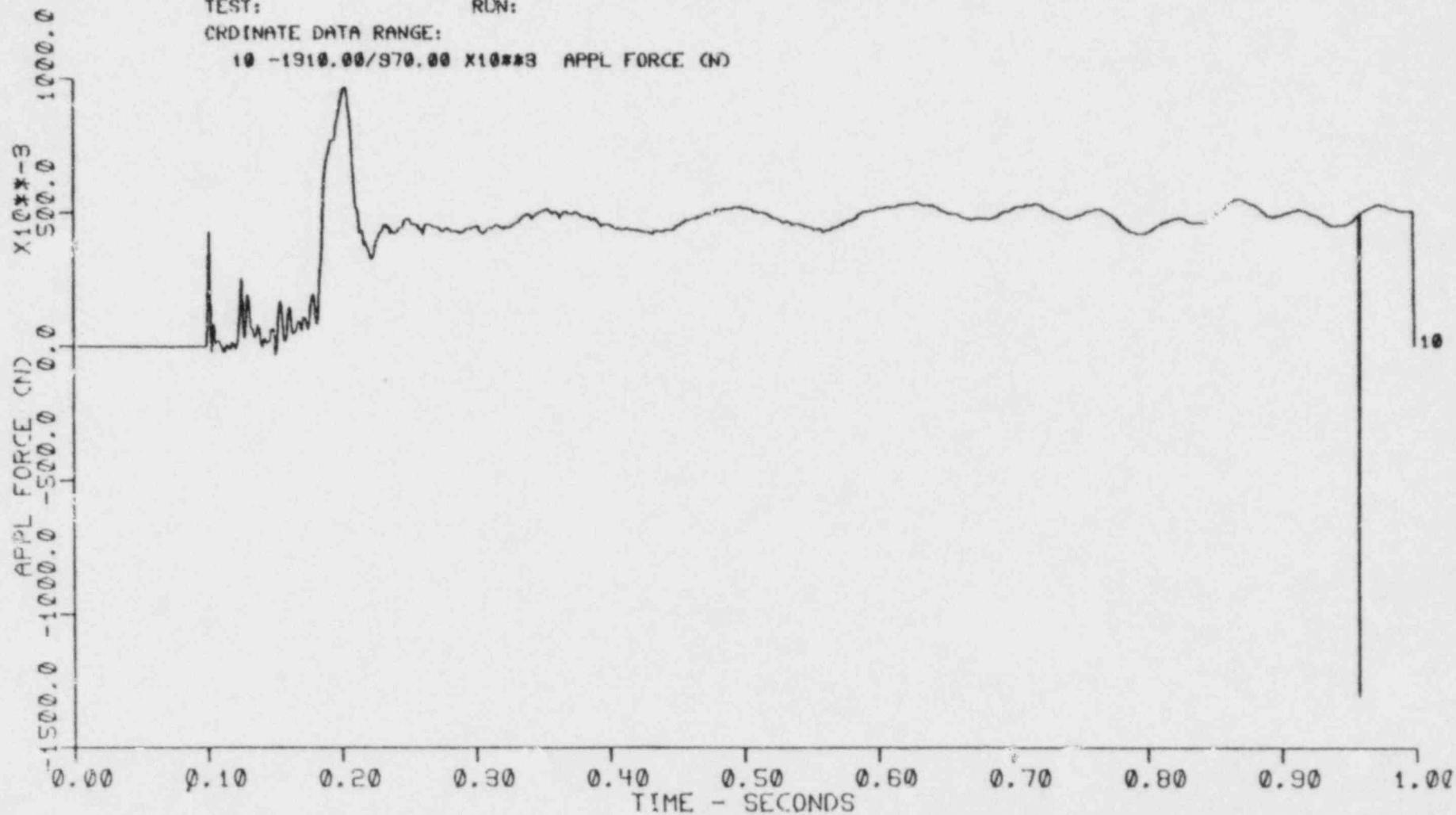
SP4AHDRSRV350

TEST:

RUN:

COORDINATE DATA RANGE:

10 -1910.00/970.00 X10\*\*3 APPL FORCE (N)



4-20

CHANNEL 10

RF0010

N

FIGURE 4.14

V60.4

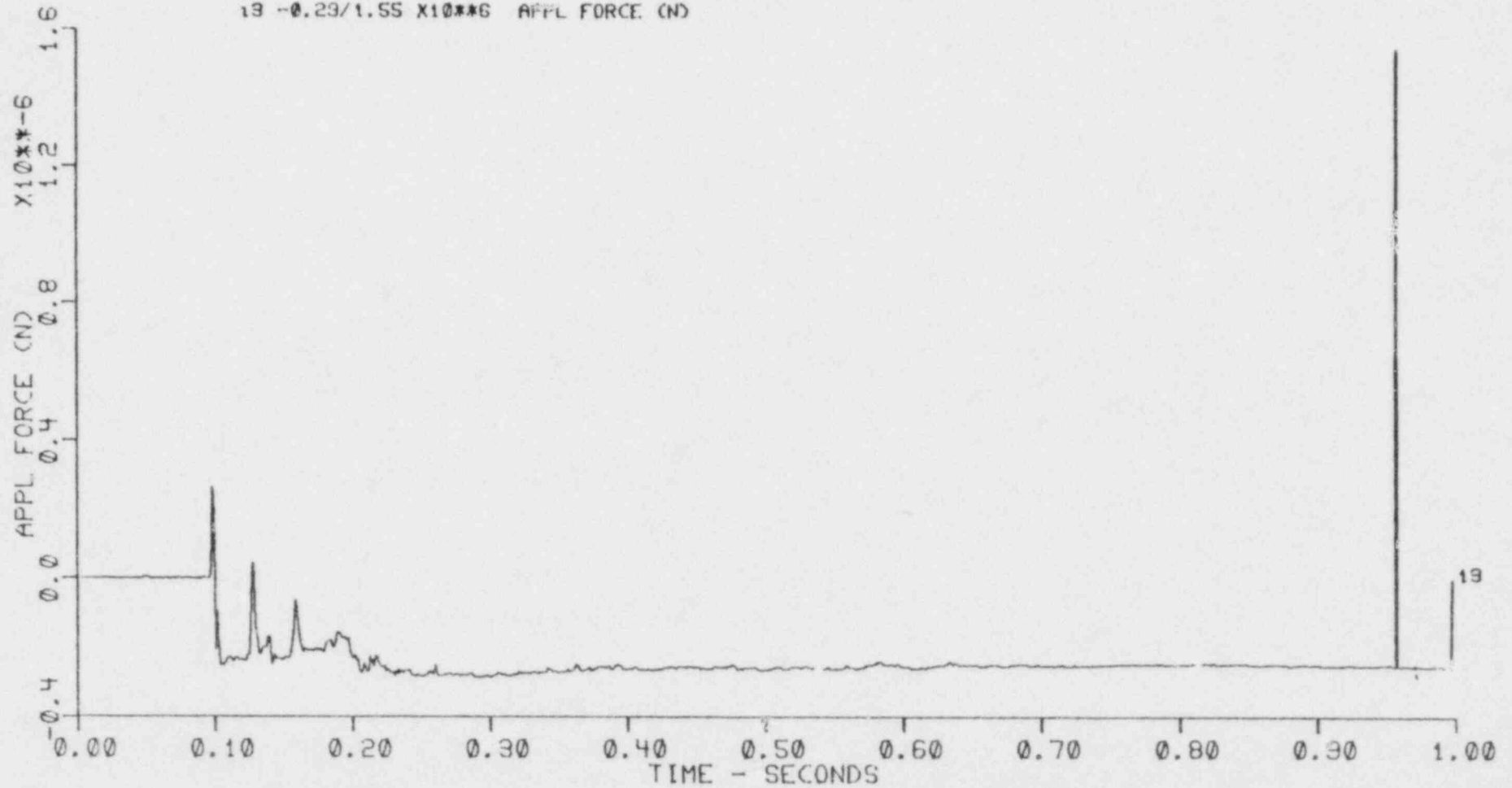
SP4AHDRSRV350

TEST:

RUN:

ORDINATE DATA RANGE:

19 -0.29/1.55 X10\*\*6 APPL FORCE (N)



CHANNEL 19

RF0013

N

FIGURE 4.15

(Problem with file information for RF0014)



TABLE 4.3

## LOCATION AND DIRECTION OF APPLIED FLUID FORCES

<u>Force Number*</u>	<u>Load Point (LP)</u>	<u>Location of Force (ANCO Node)</u>	<u>Direction of Force (Local Coordinates)**</u>
RF0001	5	11	x
RF0002	5	11	y
RF0003	5	11	z
RF0004	4	18	x
RF0005	4	18	y
RF0006	4	18	z
RF0007	3	37	x
RF0008	3	37	y
RF0009	3	37	z
RF0010	2	43	x
RF0011	2	43	y
RF0012	2	43	z
RF0013	1	49	x
RF0014	1	49	y
RF0015	1	49	z

---

\*This corresponds to the force headers of file #2.

\*\* See Figure 4.16 for the definition of these directions.



TABLE 4.4

## EXTREME VALUES OF FLUID FORCES

<u>Force Number</u>	<u>ANCO Node</u>	<u>Force Direction</u>	<u>Minimum/Maximum Value/10<sup>3</sup> (N)</u>
RF0001	11	x	-292/460
RF0002	11	y	-78/123
RF0004	18	x	-238/151
RF0005	18	y	-412/262
RF0006	18	z	-524/400
RF0007	37	x	-531/452
RF0008	37	y	-567/475
RF0010	43	x	-131/970
RF0013	49	x	-290/1550

## 5.0 STRUCTURAL DYNAMIC SIMULATION OF URL BLOWDOWN EVENT (DSP4a)

The finite element model discussed in Section 3.0 and the fluid forces presented in Section 4.0 were the basis for the response calculations required for German Standard Problem 4a (DSP4a). In performing the response calculations, several items were of concern; they were: (1) damping to be used; (2) the integration interval; and (3) nonlinear versus linear simulation methods.

No data was available on the damping of the portion of the URL pipe system involved in the DSP4a simulation. Damping data was available for the remainder of the pipe system. It typically varied from 3.0 to 6.0 percent of critical. It would be expected that the damping for the portion of the URL system not involved in DSP4a would be higher than that for the pipe leg of interest. This is because the leg of interest did not have any supports (sway braces, spring hangers, etc.) connected to it, whereas, the remainder of the system did. (The pipe supports generally increased the losses in the system.) For this reason it was decided to use damping values intermediate to the extreme values of 3.0% and 6.0%. The damping was chosen to be between 3.0% and 4.5% of critical. It should be noted that the transient response solution for DSP4a is not extremely sensitive to damping changes over a small damping domain (i.e., 3.0% to 4.0%). For this reason, choosing intermediate damping values seems to be reasonable.

The structural equations of motion were integrated using a direct integration scheme (Newmark method). The integration interval was chosen to be equal to the discretization interval used for digitizing the DSP4 data ( $\Delta t = 0.0002$  second). With this time interval and using 16 time points per cycle, it is possible to define a transient signal of up to 312.5 Hz. The fluid pressure, for DSP4 type events, generally has its major frequency content in the 50 Hz to 100 Hz range. Hence, the chosen integration

interval should be more than adequate (Nyquist sampling theorem says that a  $\Delta t$  of 0.0002 seconds is sufficient to detect transients up to 2500 Hz while experience indicates that such a  $\Delta t$  will certainly be sufficient for transients up to 1200 Hz).

The proportional damping coefficients  $\alpha$  and  $\beta$  ( $C = \alpha M + \beta K$ , where  $C$ ,  $M$ , and  $K$  are the damping, mass, and stiffness matrices, respectively) were chosen to be 1.20 and  $6.35 \times 10^{-5}$ , respectively. This gives an equivalent modal damping of 4.5% at 5 Hz and 100 Hz and a minimum damping of 3.0% between 5 Hz and 100 Hz.

In simulating the structural dynamic event, a linear analysis approach was selected. This decision was made because it was not known if the pipe would experience nonlinear deformation; hence, a standard approach dictated that a linear analysis be performed first, and, only if necessary, a nonlinear analysis would be performed (this would not be done for this present task).

The linear analysis was performed and the plotted results shown in Figures 5.1 to 5.42. Table 5.1 gives the minimum and maximum values for the structural response quantities specified by DSP4a. Appendix C lists the computer code that was written to process the EASE2 output. It was used to compute additional stress information and perform the necessary coordinate transformations of the displacement and acceleration results.

FIGURE 5.1  
V60.4

SP4AHDRSRV350

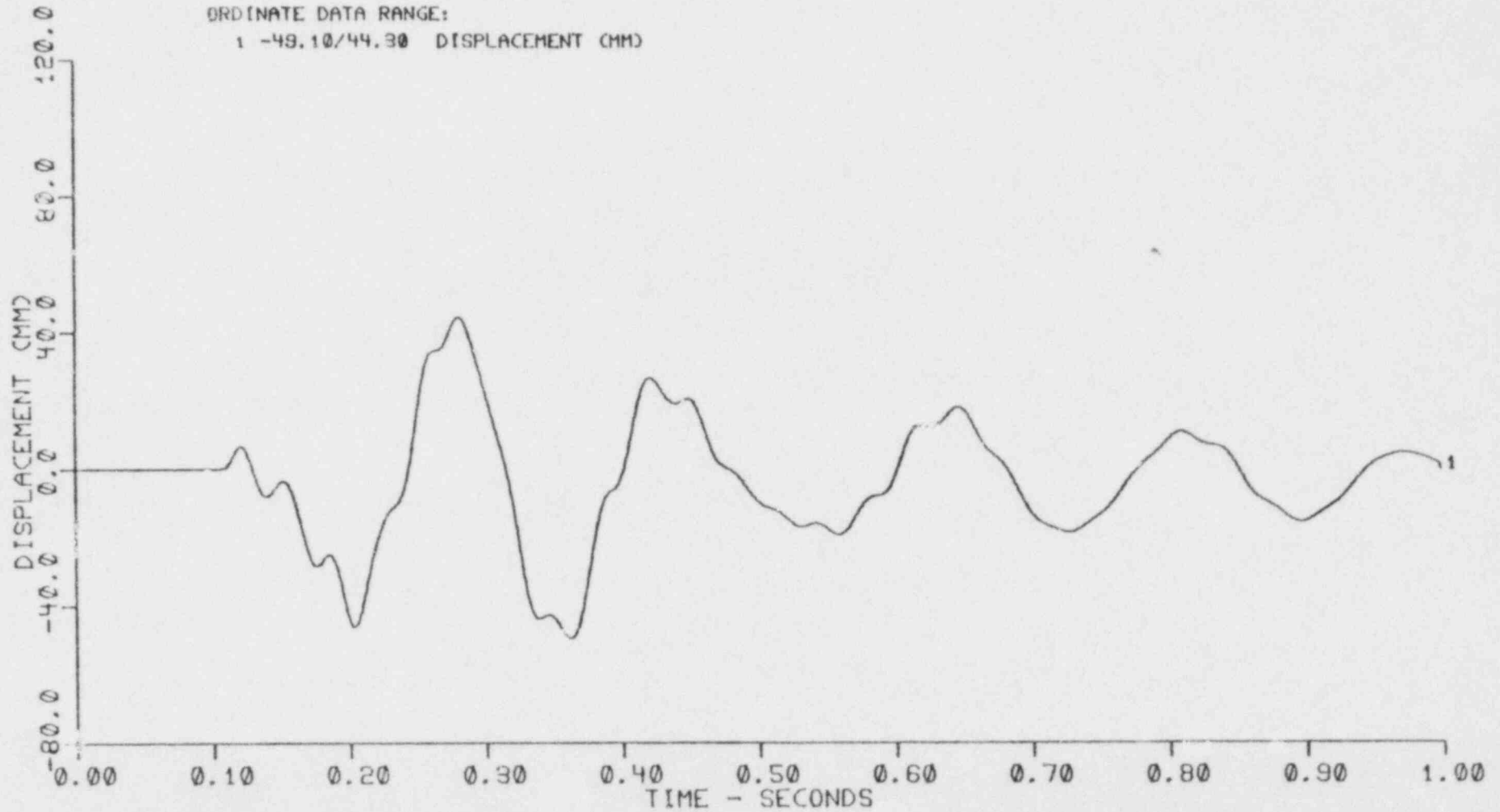
TEST:

RUN:

ORDINATE DATA RANGE:

1 -49.10/44.30 DISPLACEMENT (MM)

S-3



CHANNEL 1

RS2201

MM

FIGURE 5.2

V60.4

SP4AHDRSRV350

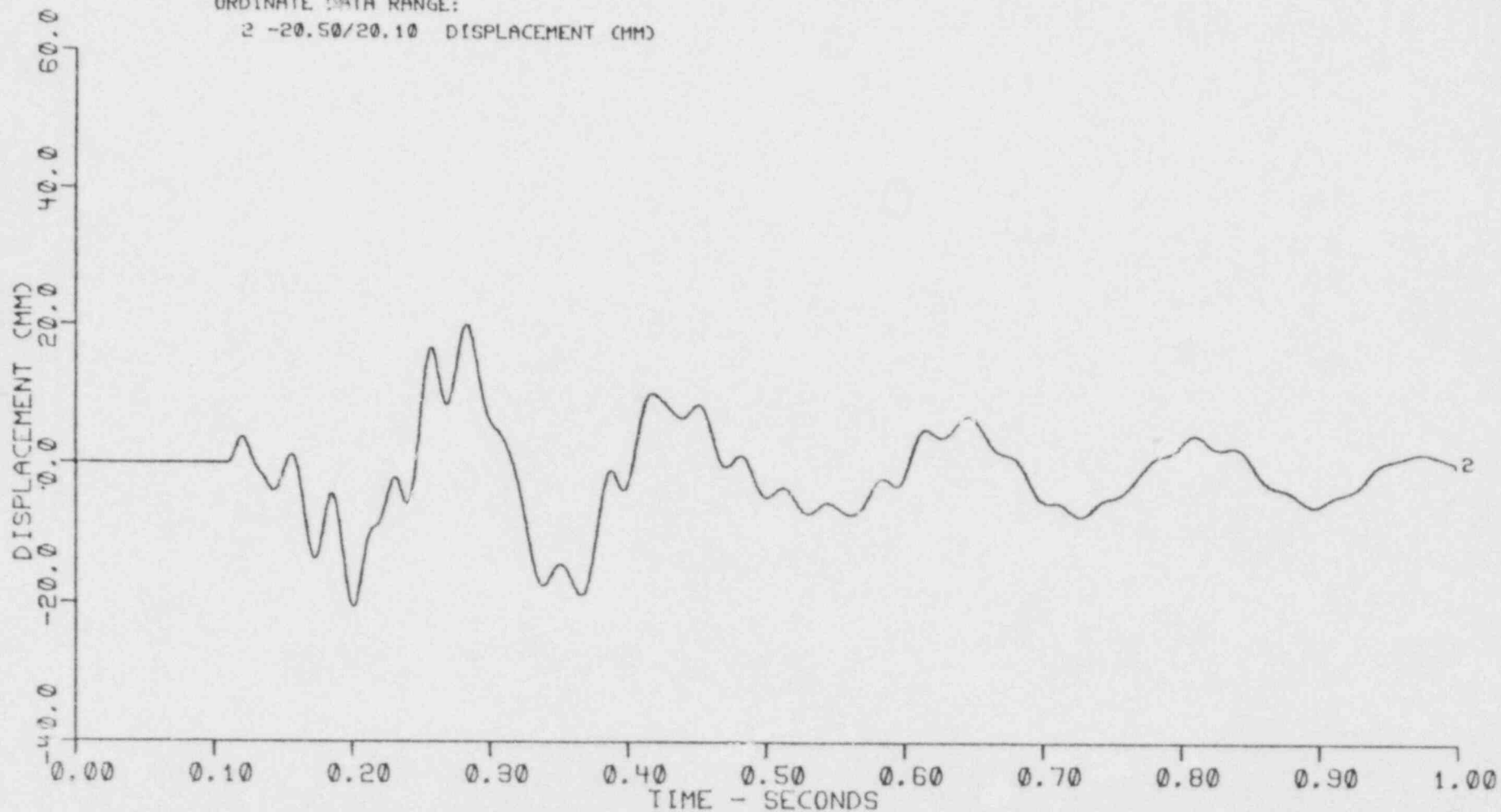
TEST:

RUN:

ORDINATE DATA RANGE:

2 -20.50/20.10 DISPLACEMENT (MM)

S-4



CHANNEL 2

RS2202

MM

FIGURE 5.3

V60.4

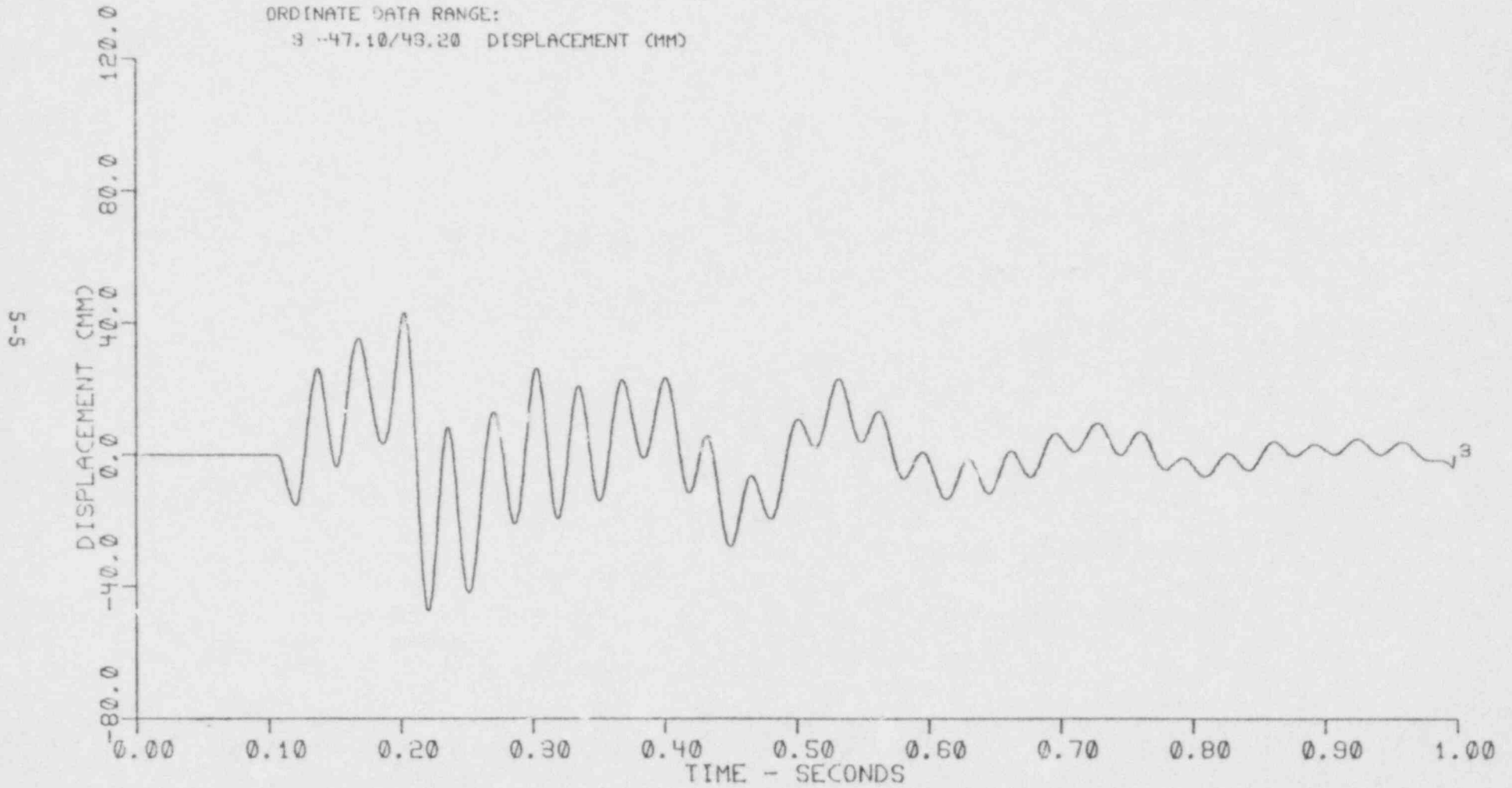
SP4AHDRSRV350

TEST:

RUN:

ORDINATE DATA RANGE:

3 -47.10/43.20 DISPLACEMENT (MM)



CHANNEL 3

RS2203

MM



FIGURE 5.4  
V60.4

SP4AHDRSRV350

TEST:

RUN:

ORDINATE DATA RANGE:

4 -87.20/66.70 DISPLACEMENT (MM)

9-5

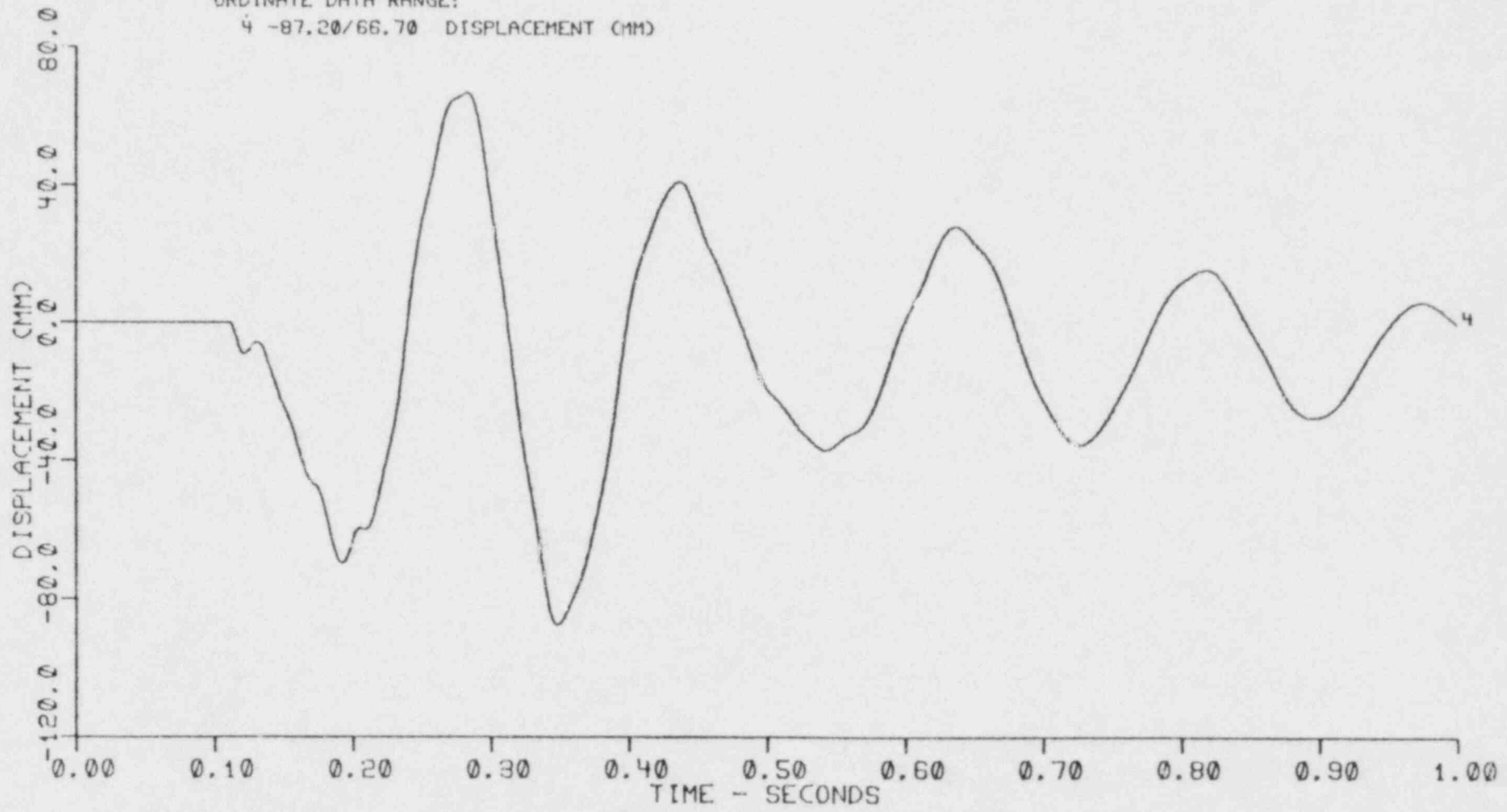


FIGURE 5.5

V60.4

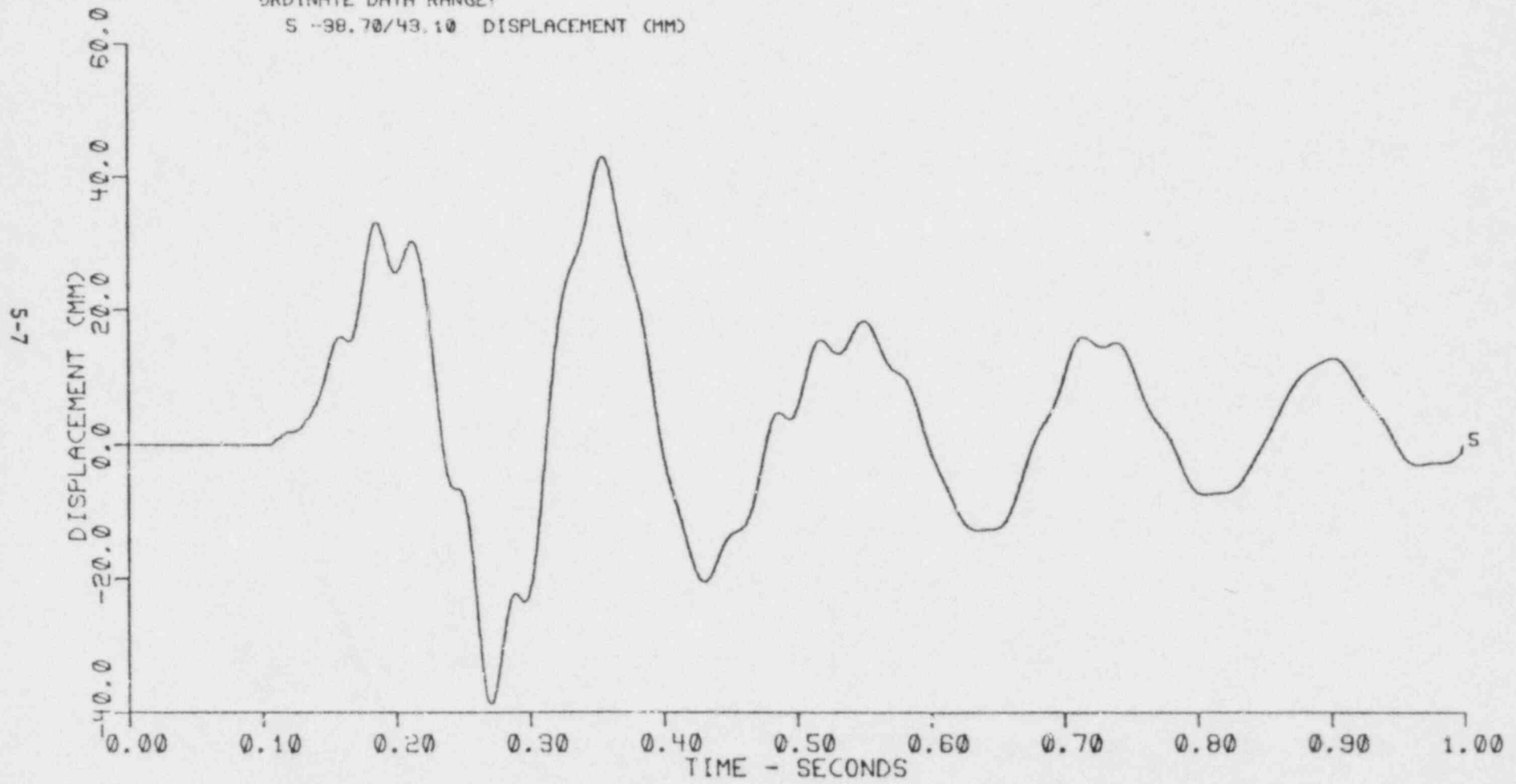
SP4AHDRSRV350

TEST:

RUN:

ORDINATE DATA RANGE:

S -38.70/43.10 DISPLACEMENT (MM)



CHANNEL 5

RS2205

MM

FIGURE 5.6  
V60.4

SP4AHDRSRV350

TEST:                      RUN:  
ORDINATE DATA RANGE:  
6 -50.10/41.30 DISPLACEMENT (MM)

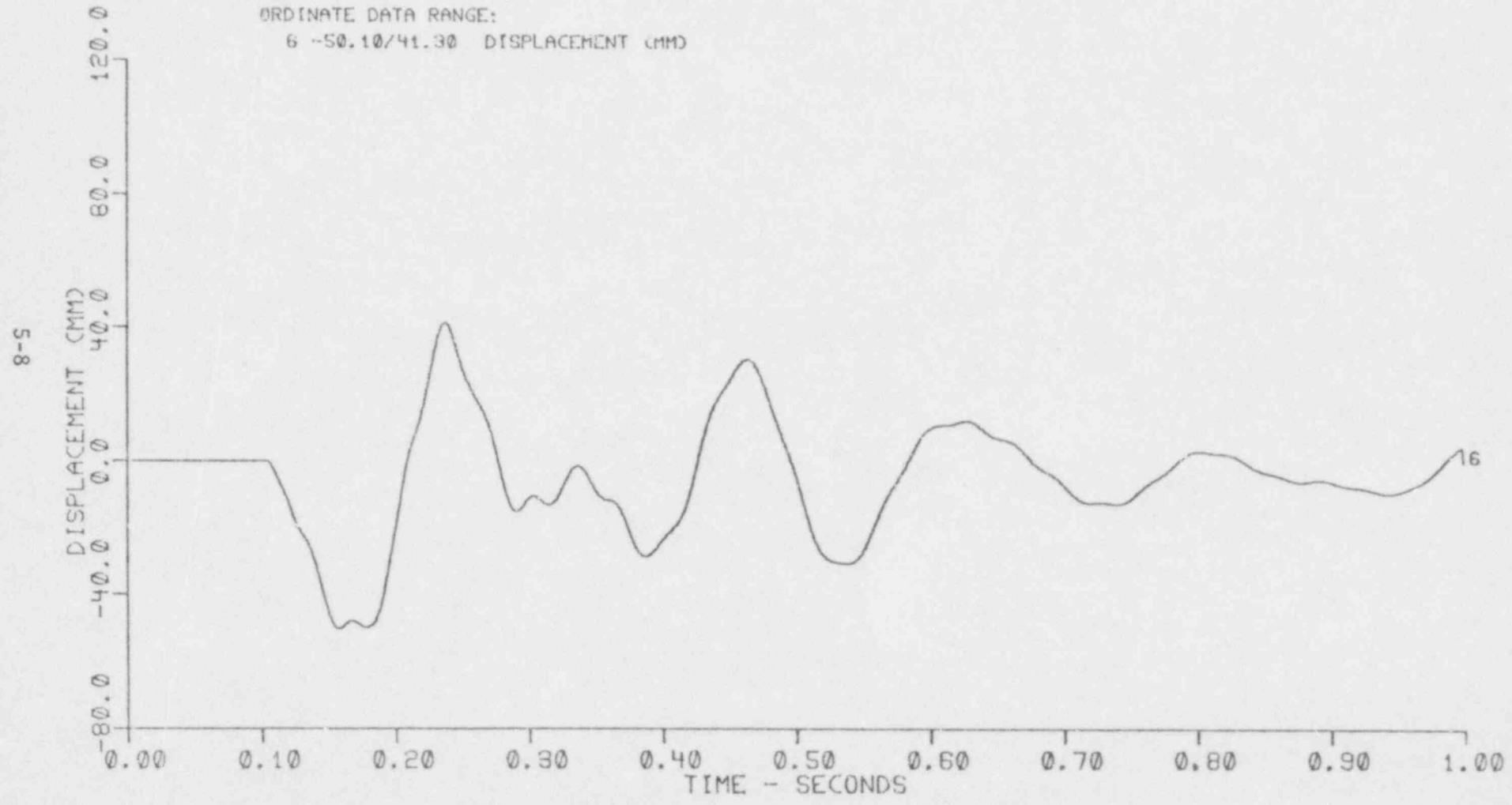


FIGURE 5.7

V60.4

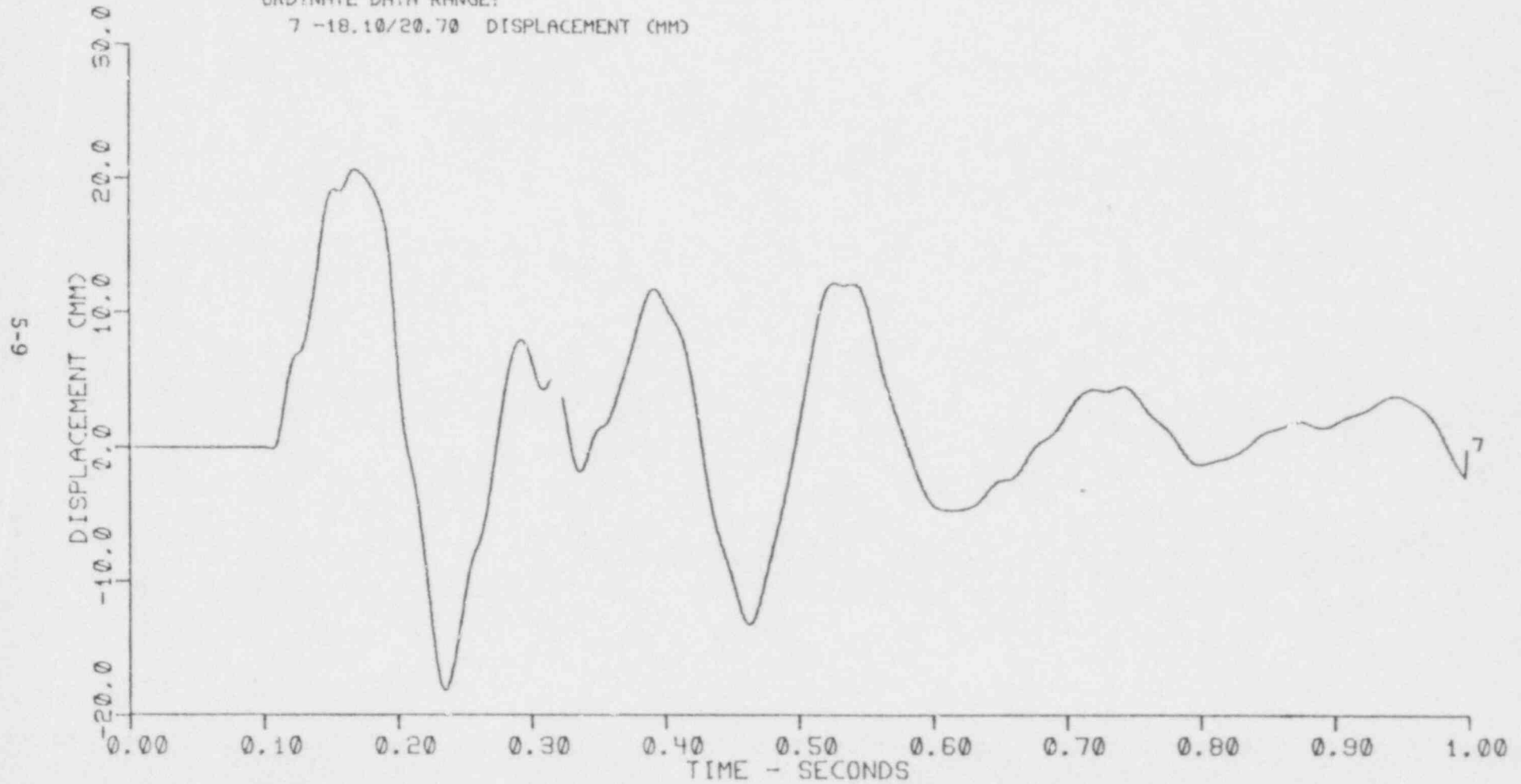
SP4AHDRSRV350

TEST:

RUN:

ORDINATE DATA RANGE:

7 -18.10/20.70 DISPLACEMENT (MM)



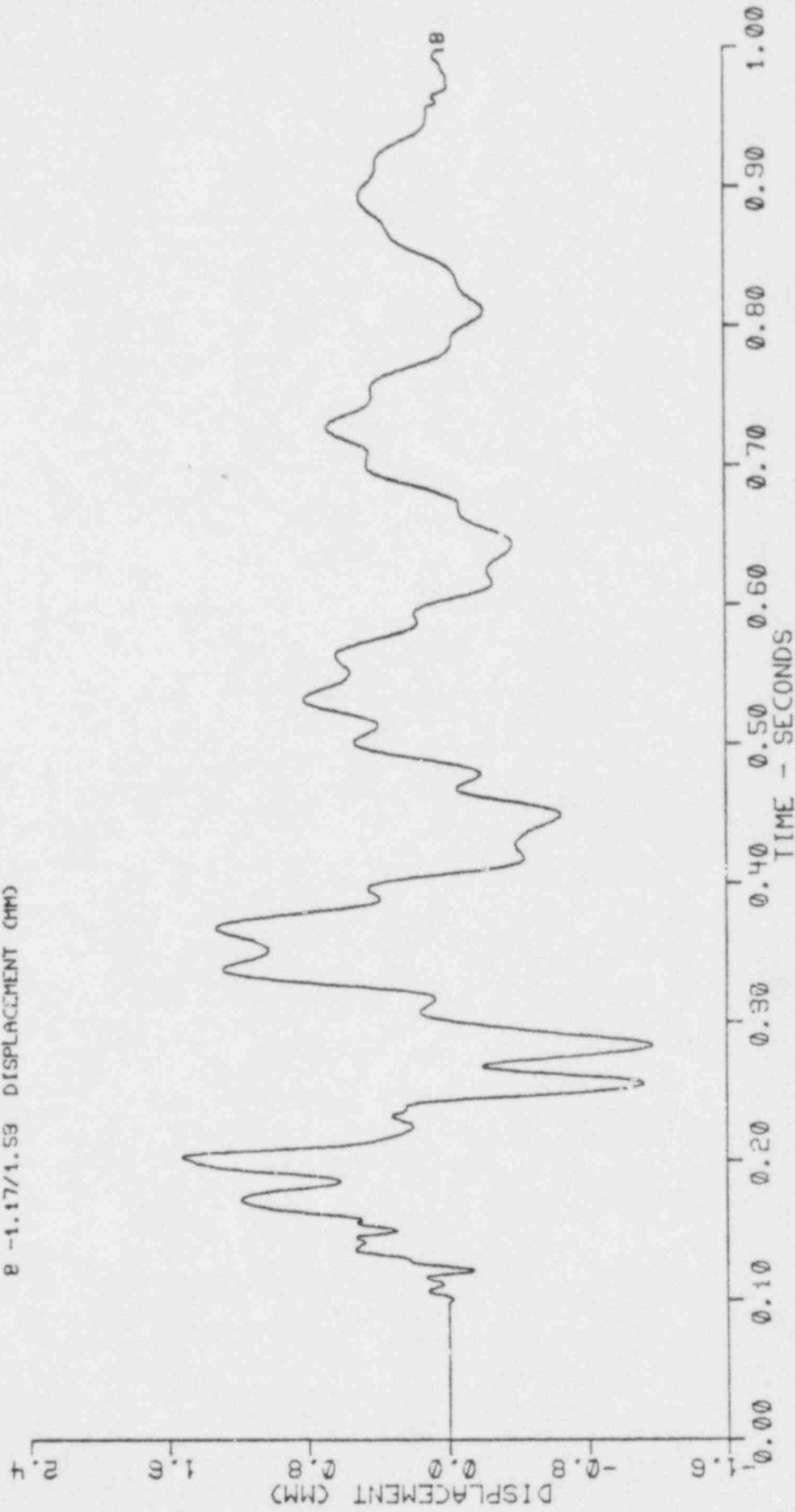
CHANNEL 7

SS4004

MM

FIGURE 5.8  
V60.4 SP4HDRSRV350

TEST: RUN:  
ORDINATE DATA RANGE:  
B -1.17/1.53 DISPLACEMENT (MM)



CHANNEL 8 554005 MM

FIGURE 5.9

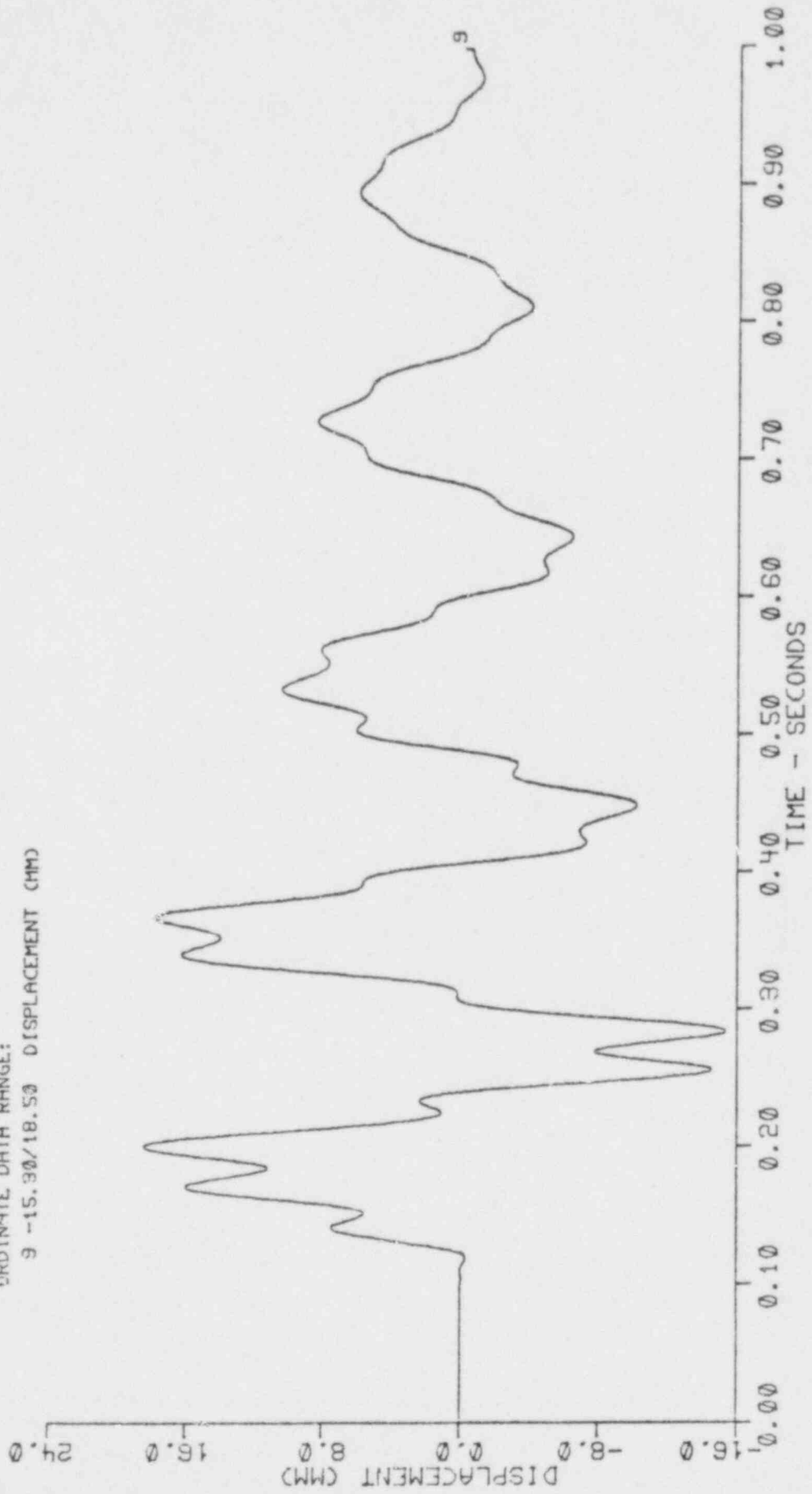
V60.4

SP4AHDRSRV350

TEST: RUN:

ORDINATE DATA RANGE:

9 -15.30/18.50 DISPLACEMENT (MM)



CHANNEL 9 SS4006 MH

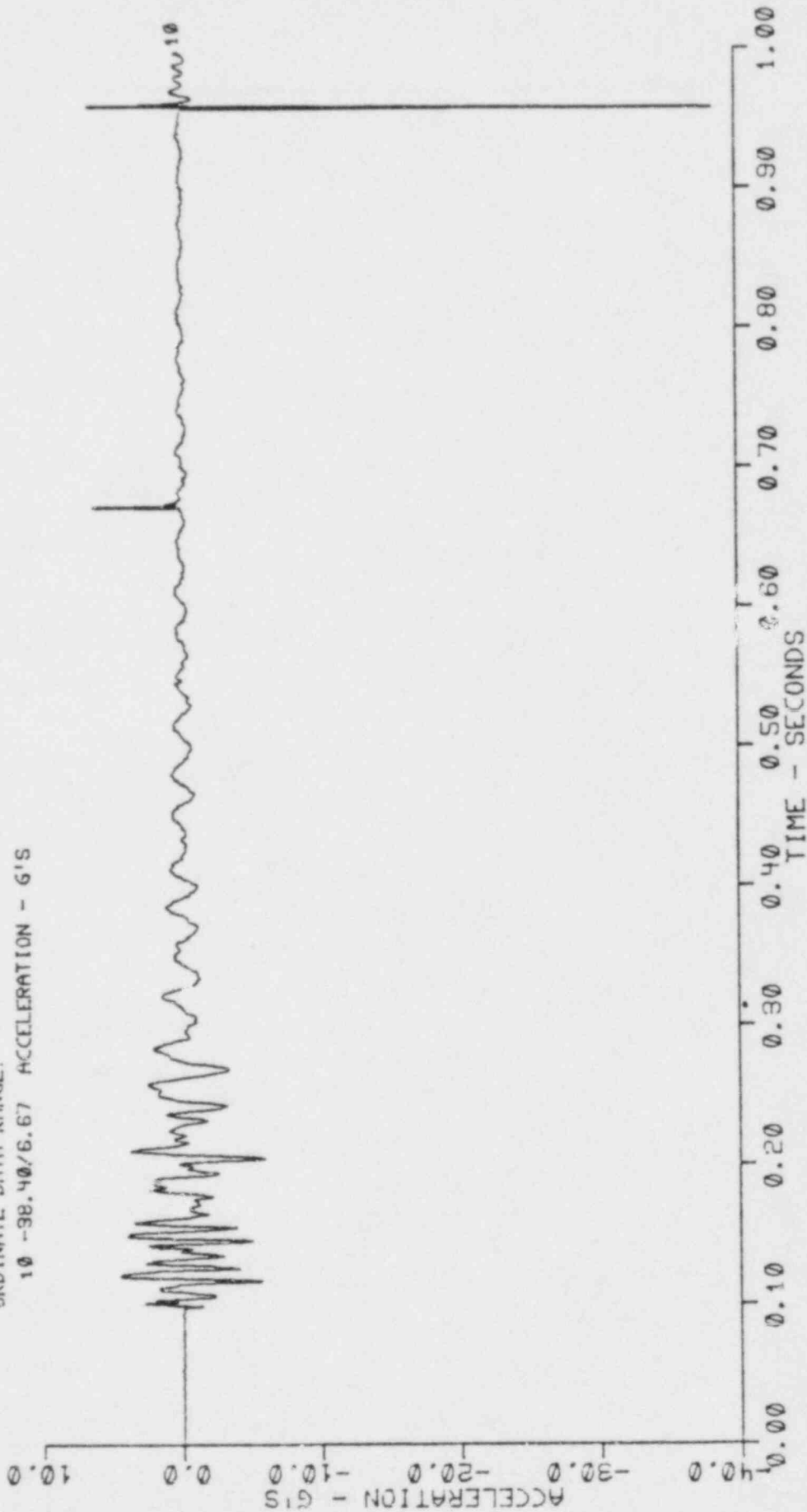
FIGURE 5.10

V60.4 SP4AHDRSRV350

TEST: RUN:

ORDINATE DATA RANGE:

10 -38.40/6.67 ACCELERATION - G'S



CHANNEL 10 SS4001 G

FIGURE 5.11

V60.4

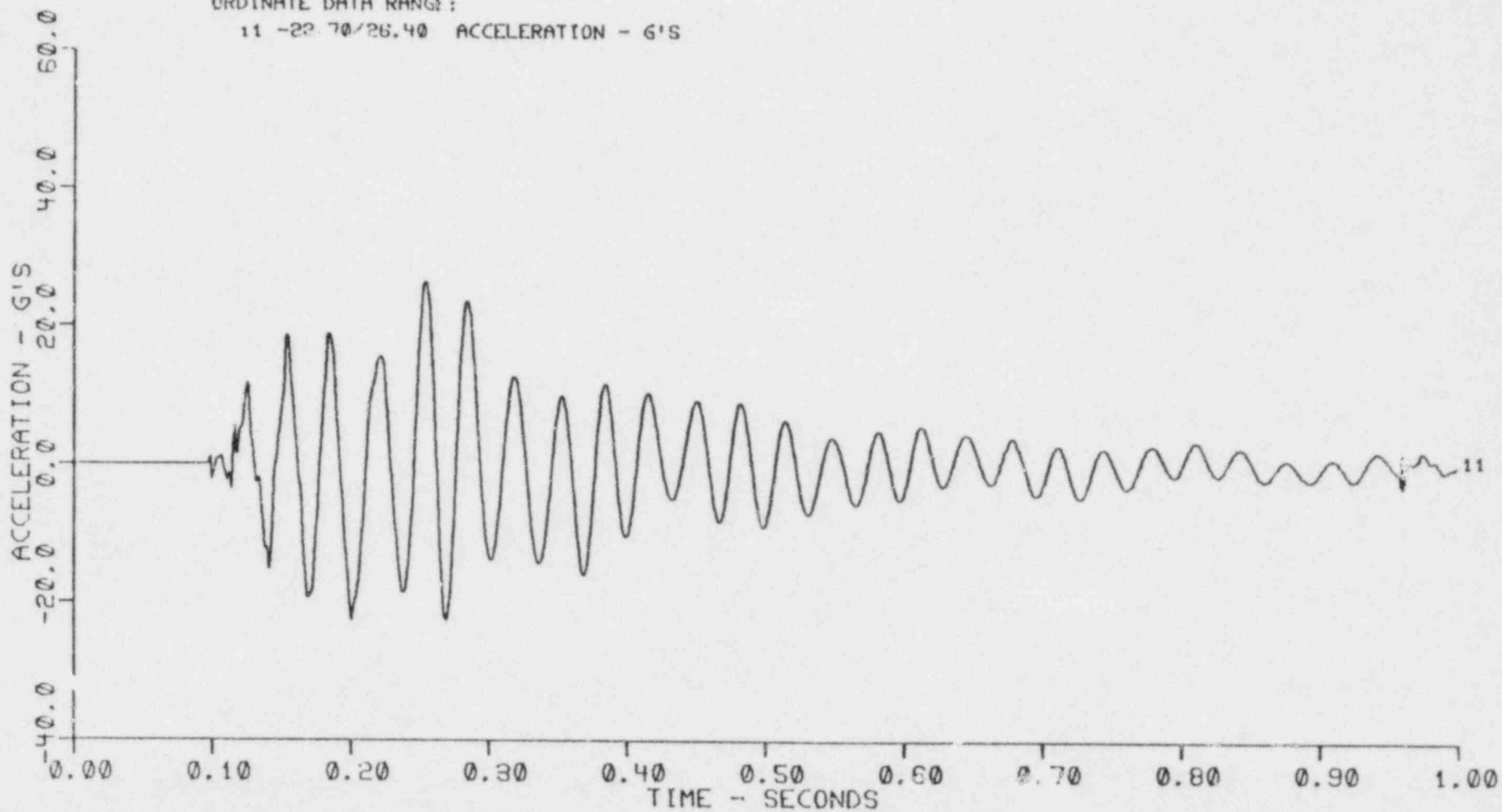
SP4AHDRSRV350

TEST:

RUN:

ORDINATE DATA RANGE:

11 -22.70/26.40 ACCELERATION - G'S



CHANNEL 11

SS4002

G

5-13



FIGURE 5.12  
V60.4

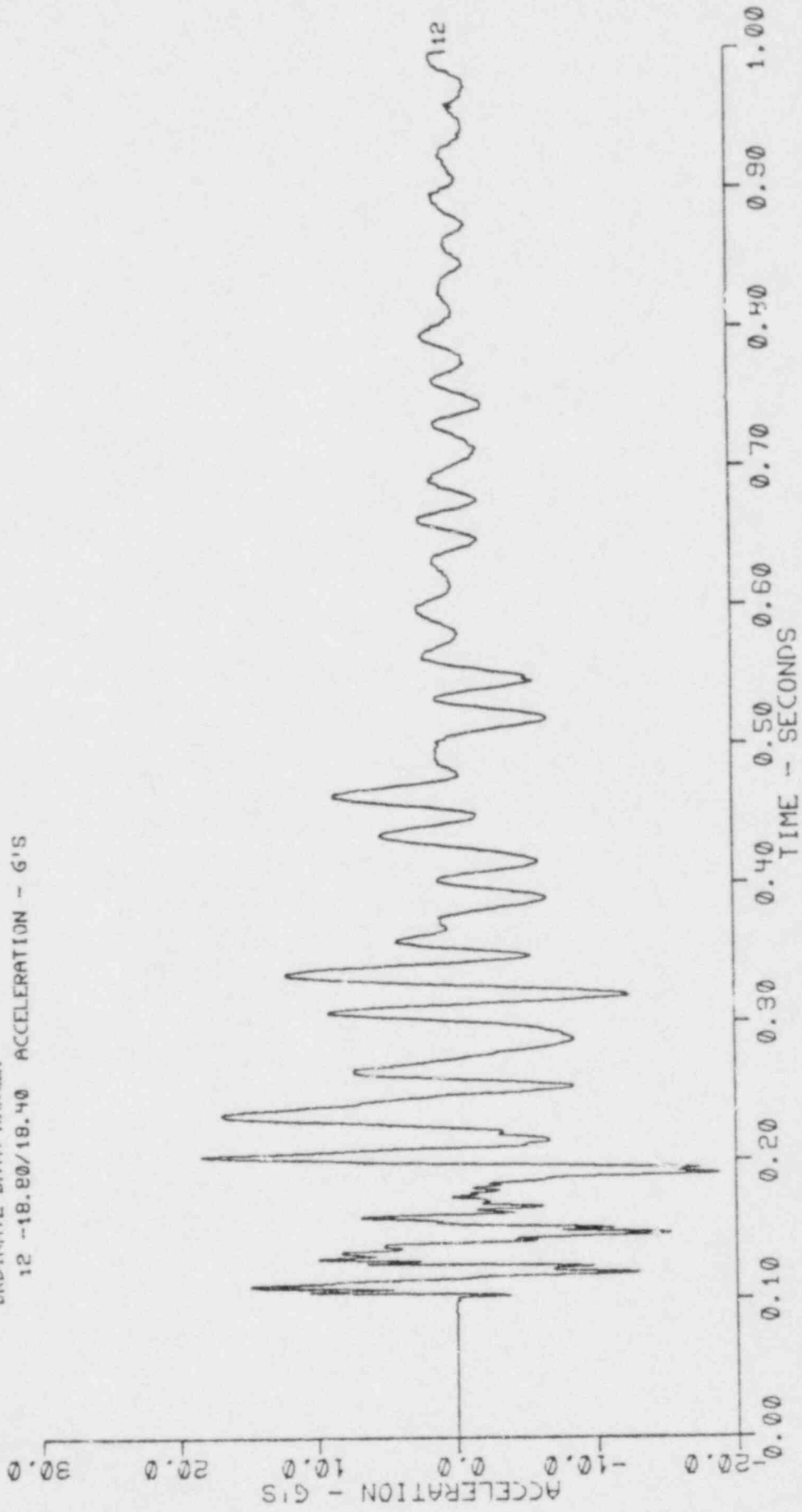
SP4AHDRSRV350

RUN:

TEST:

ORDINATE DATA RANGE:

12 -18.80/18.40 ACCELERATION - G'S



CHANNEL 12 SS4603 G

FIGURE 5.13

V60.4

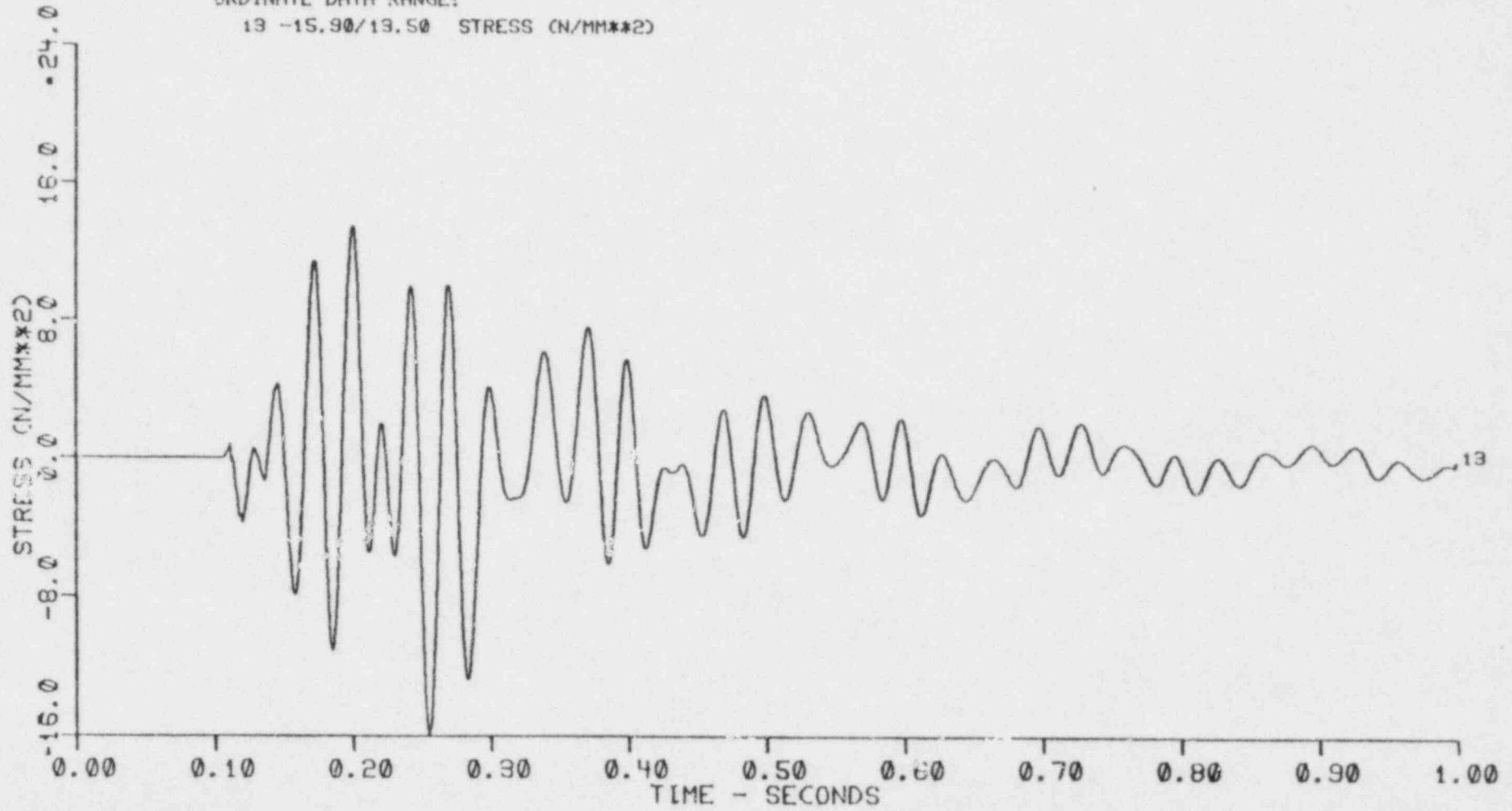
SP4AHDRSRV350

TEST:

RUN:

ORDINATE DATA RANGE:

13 -15.90/13.50 STRESS (N/MM\*\*2)



CHANNEL 13

001010

N/MM2

5-15

FIGURE 5.14

(Problem with File Information for RK 2010)

FIGURE 5.15

VS0.4

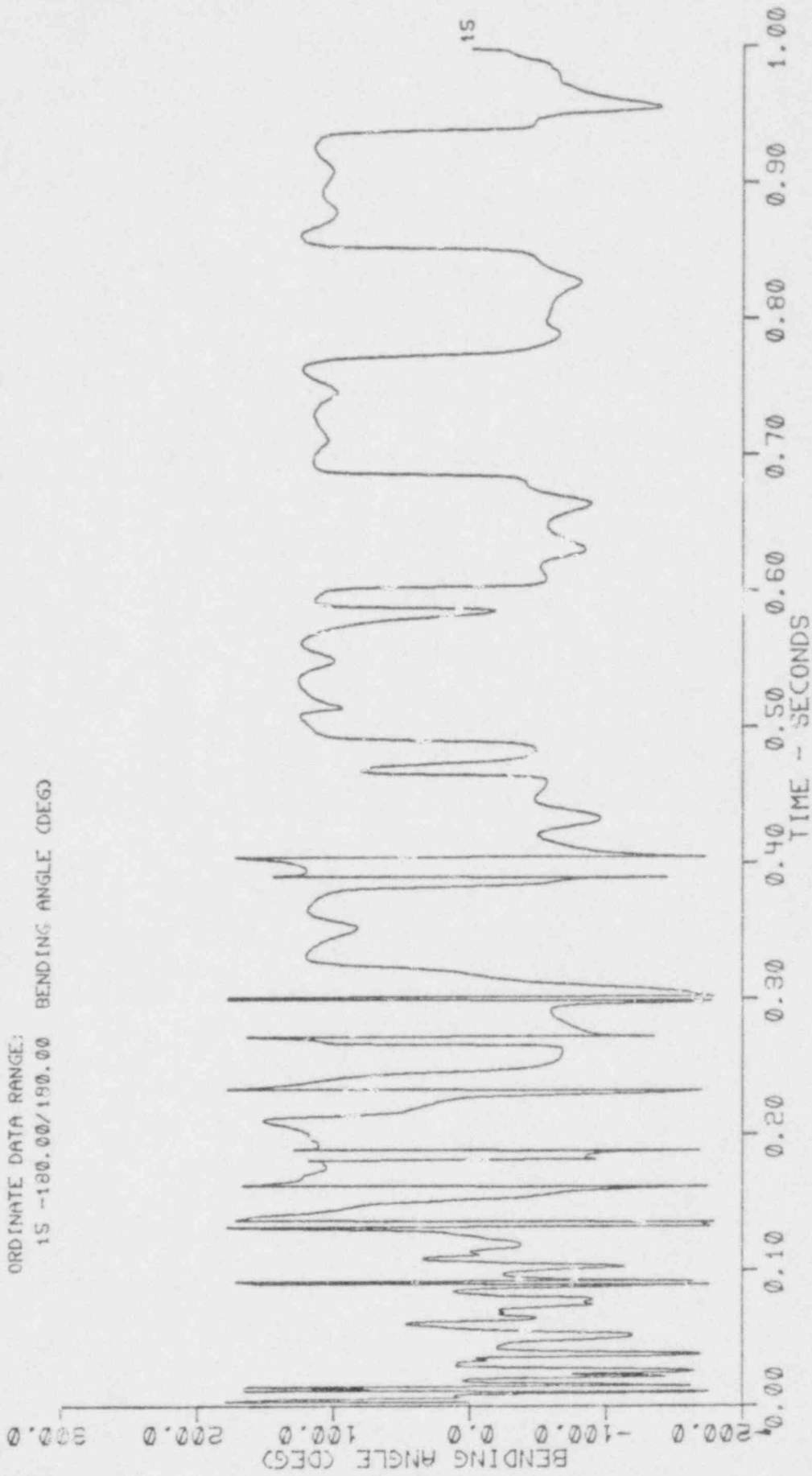
SP4HDRSRV350

RUN:

TEST:

ORDINATE DATA RANGE:

15 -180.00/180.00 BENDING ANGLE (DEG)



CHANNEL 15 RK2210 GRAD

FIGURE 5.16  
V60.4

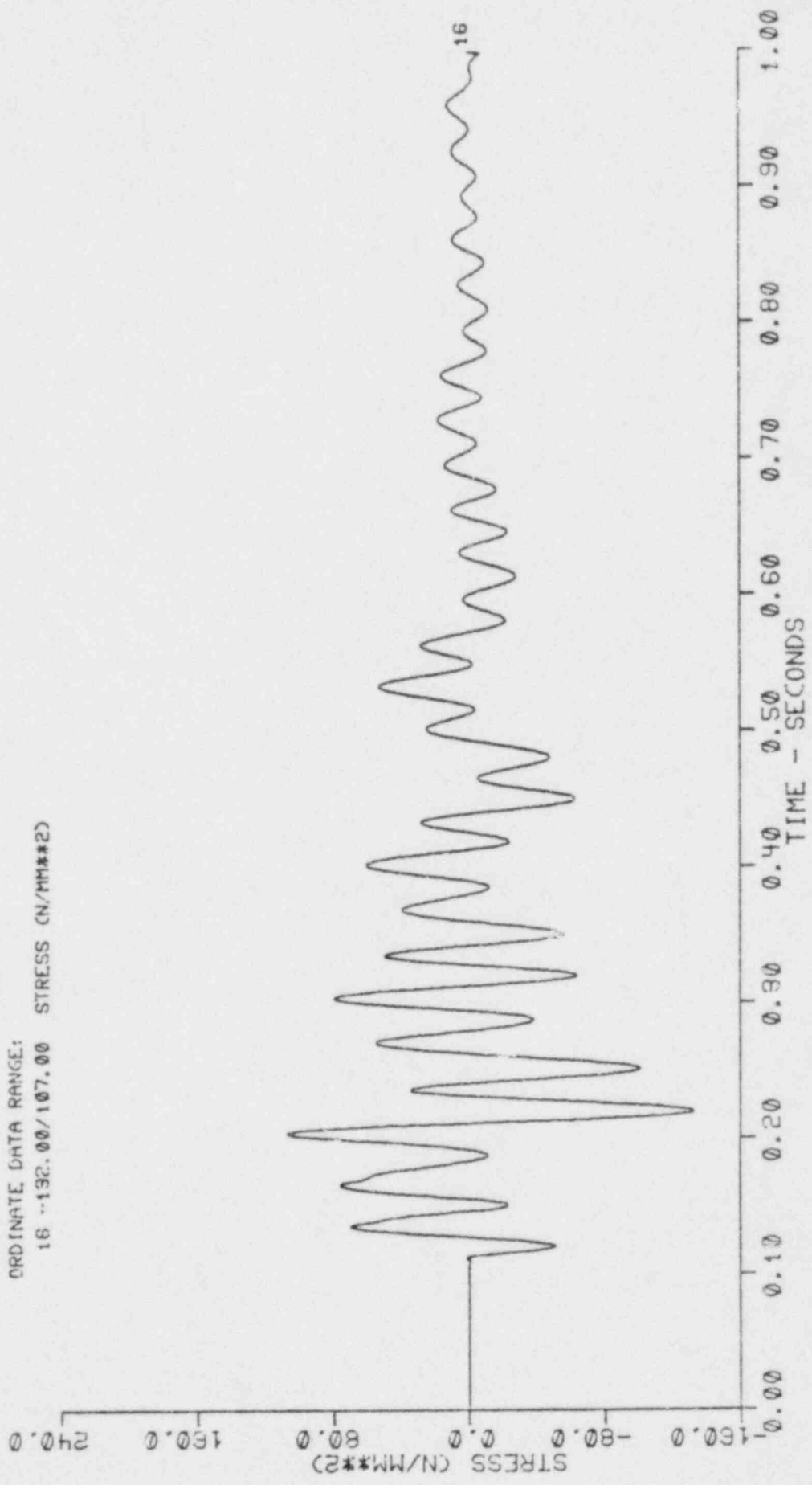
SP4AHDRSRV350

RUN:

TEST:

ORDINATE DATA RANGE:

16 --132.00/107.00 STRESS (N/MM\*\*2)



CHANNEL 16 RK3010 N/MM2

FIGURE 5.17  
(RK 4110 not computed)

FIGURE 5.18

(RK 5010 not computed)

FIGURE 5.19

VE0.4

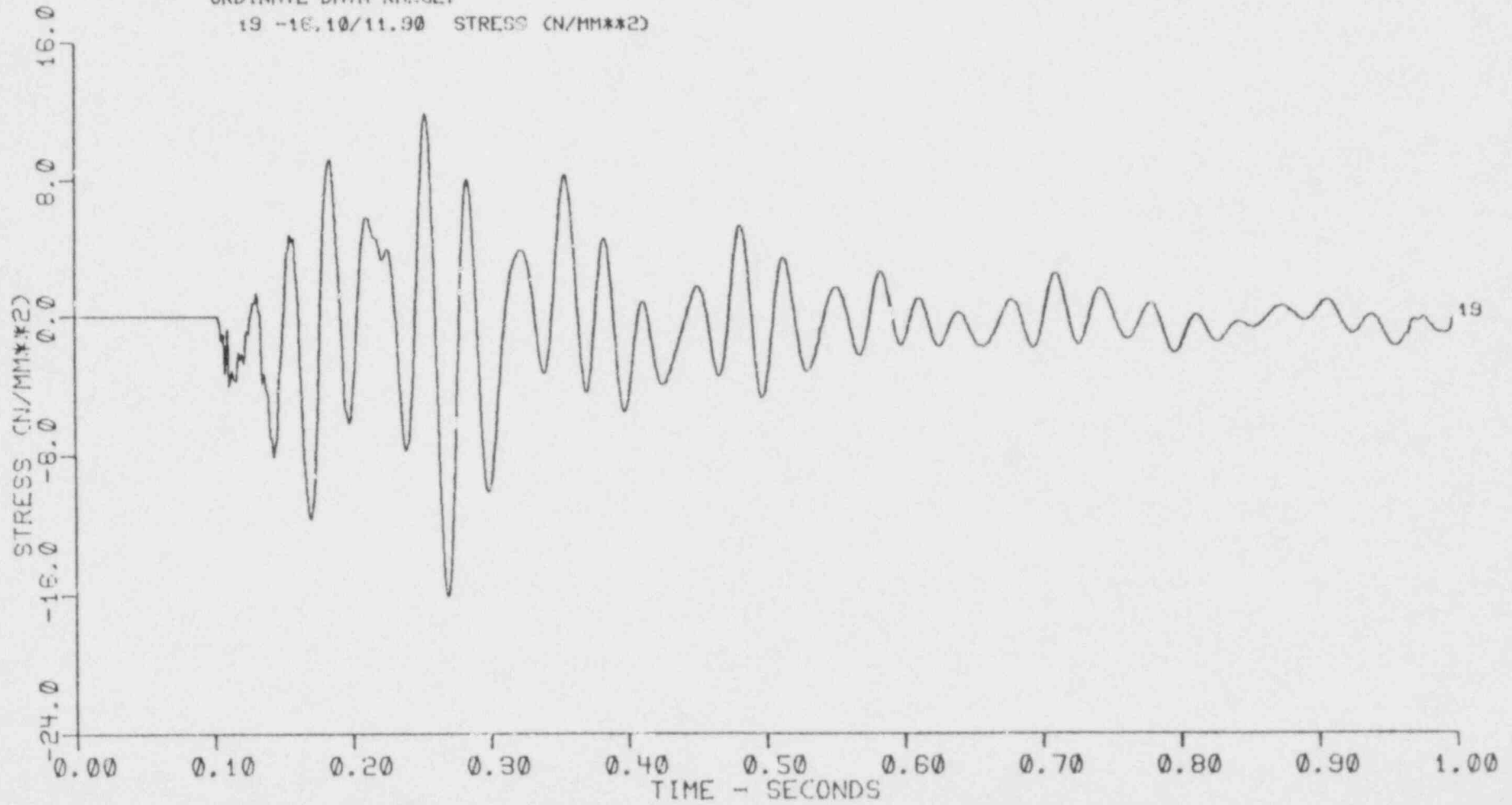
SP4AHDRSRV350

TEST:

RUN:

ORDINATE DATA RANGE:

19 -16.10/11.90 STRESS (N/MM\*\*2)



CHANNEL 19

RK1011

N/MM2



FIGURE 5.20

(Problem with File Information for RK 2011)

FIGURE 5.21

V6C.4

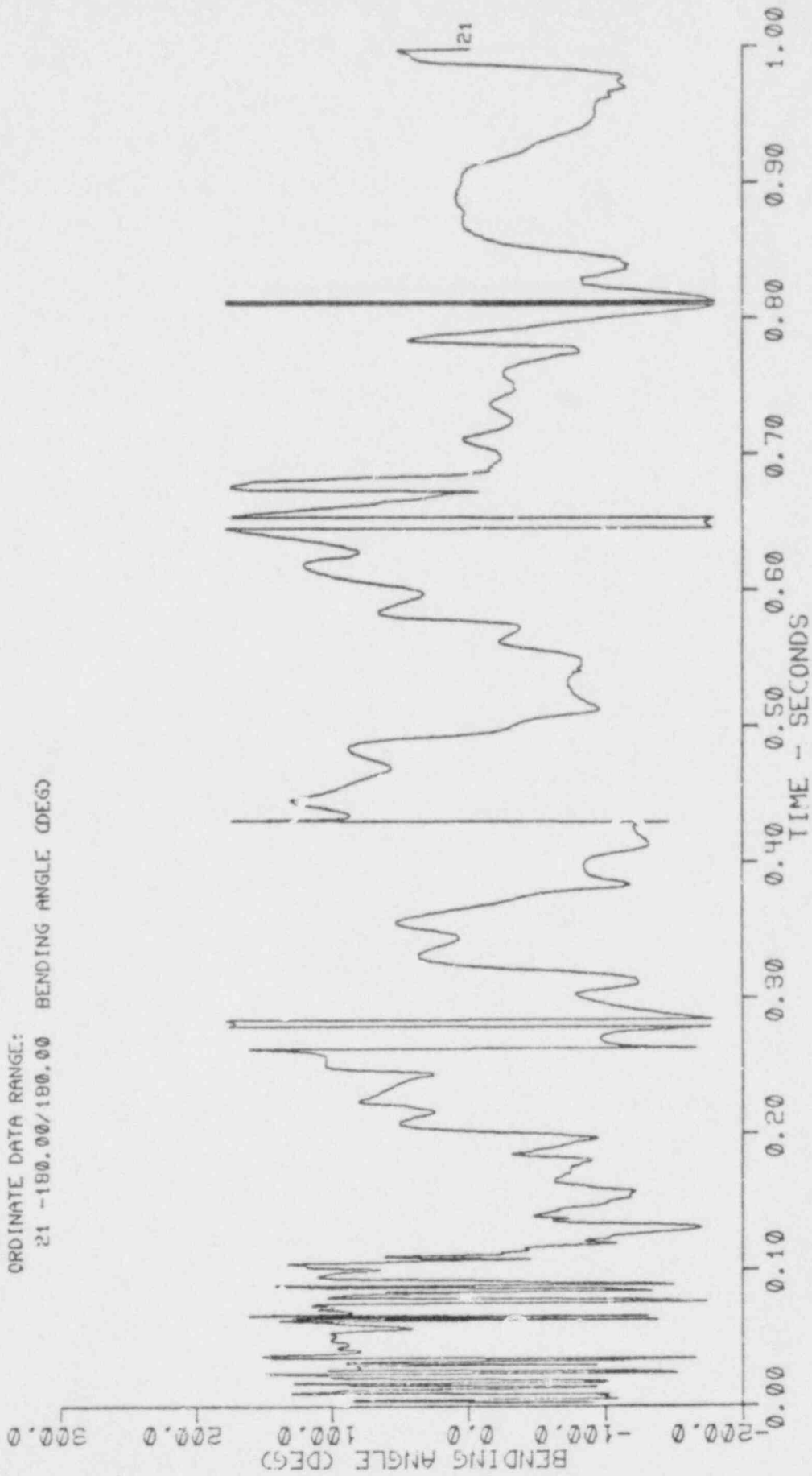
SP4AHDRSRV350

RUN:

TEST:

ORDINATE DATA RANGE:

21 -180.00/180.00 BENDING ANGLE (DEG)



CHANNEL 21

RK2211

GRAB

FIGURE 5.22

V60.4

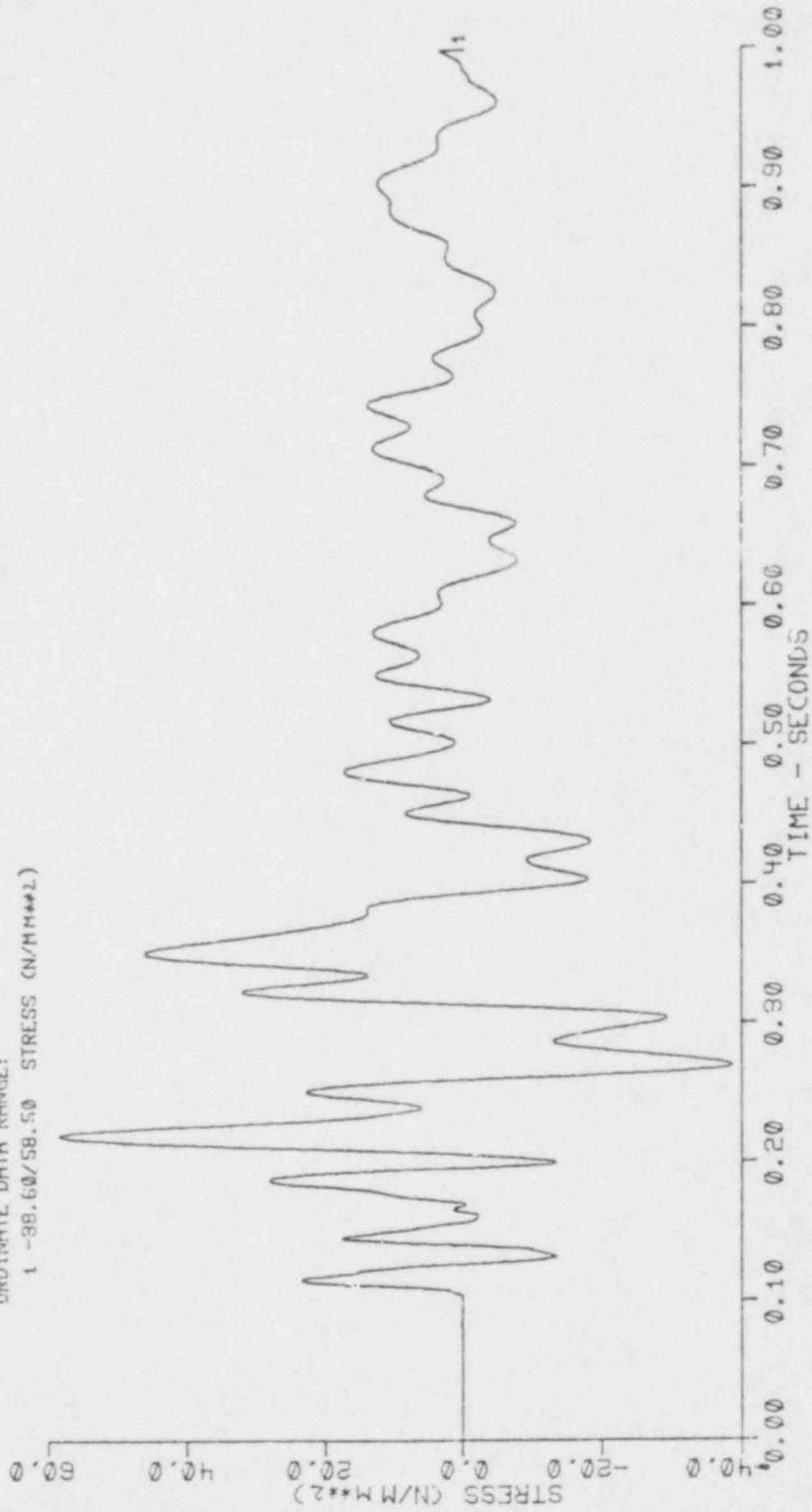
SP4AHDRSRV350

RUN:

TEST:

ORDINATE DATA RANGE:

1 -38.60/58.50 STRESS (N/MM\*\*2)



CHANNEL 1 RK3011 N/Hz

FIGURE 5.23  
(RK 4111 not computed)

FIGURE 5.24  
(RK 5011 not computed)

FIGURE 5.25

V60.4

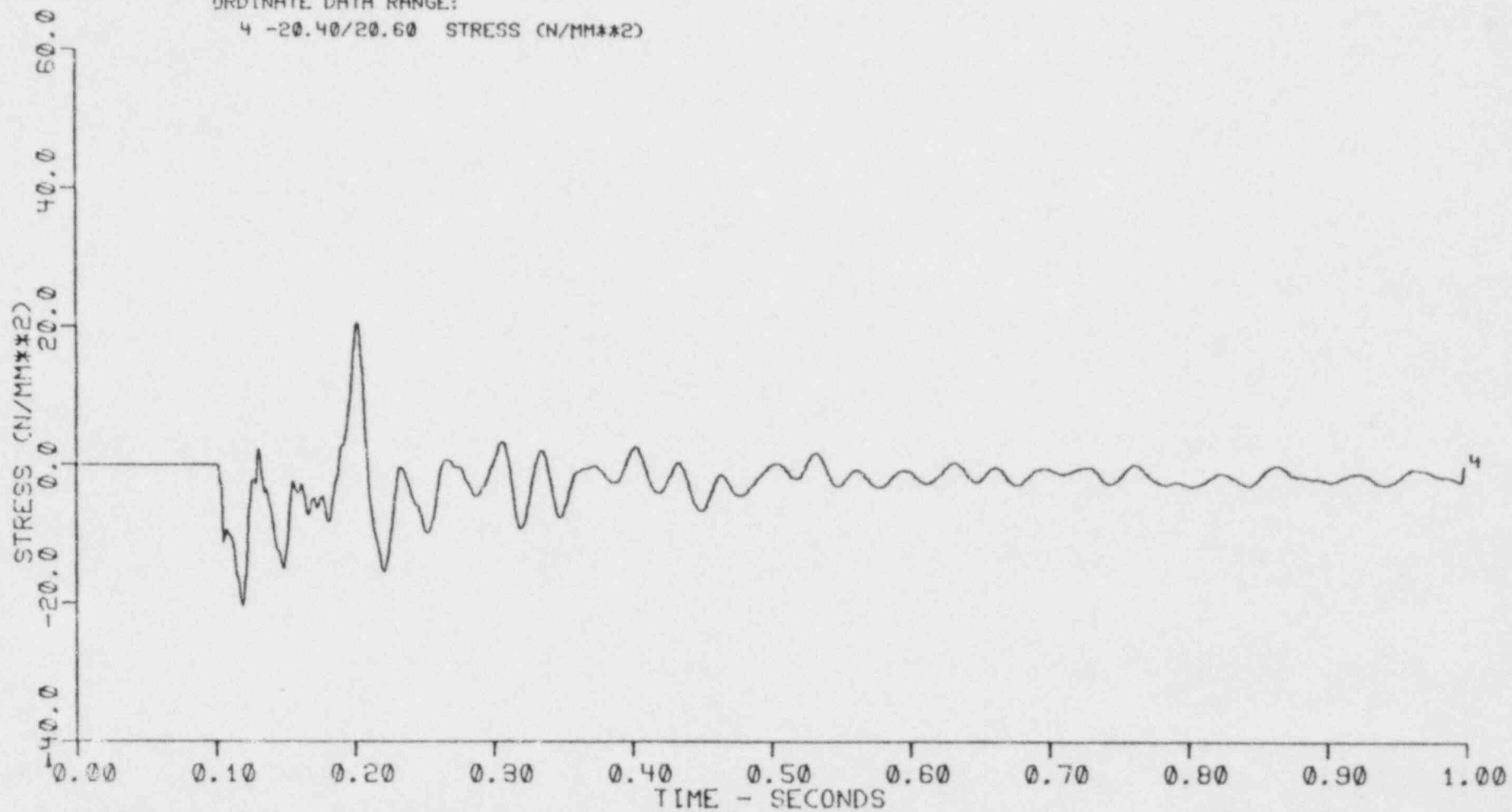
SP4AHDRSRV350

TEST:

RUN:

ORDINATE DATA RANGE:

4 -20.40/20.60 STRESS (N/MM\*\*2)



S-27

CHANNEL 4

RK1012

N/MM2

FIGURE 5.26

(Problem with file information for RK 2012)

FIGURE 5.27

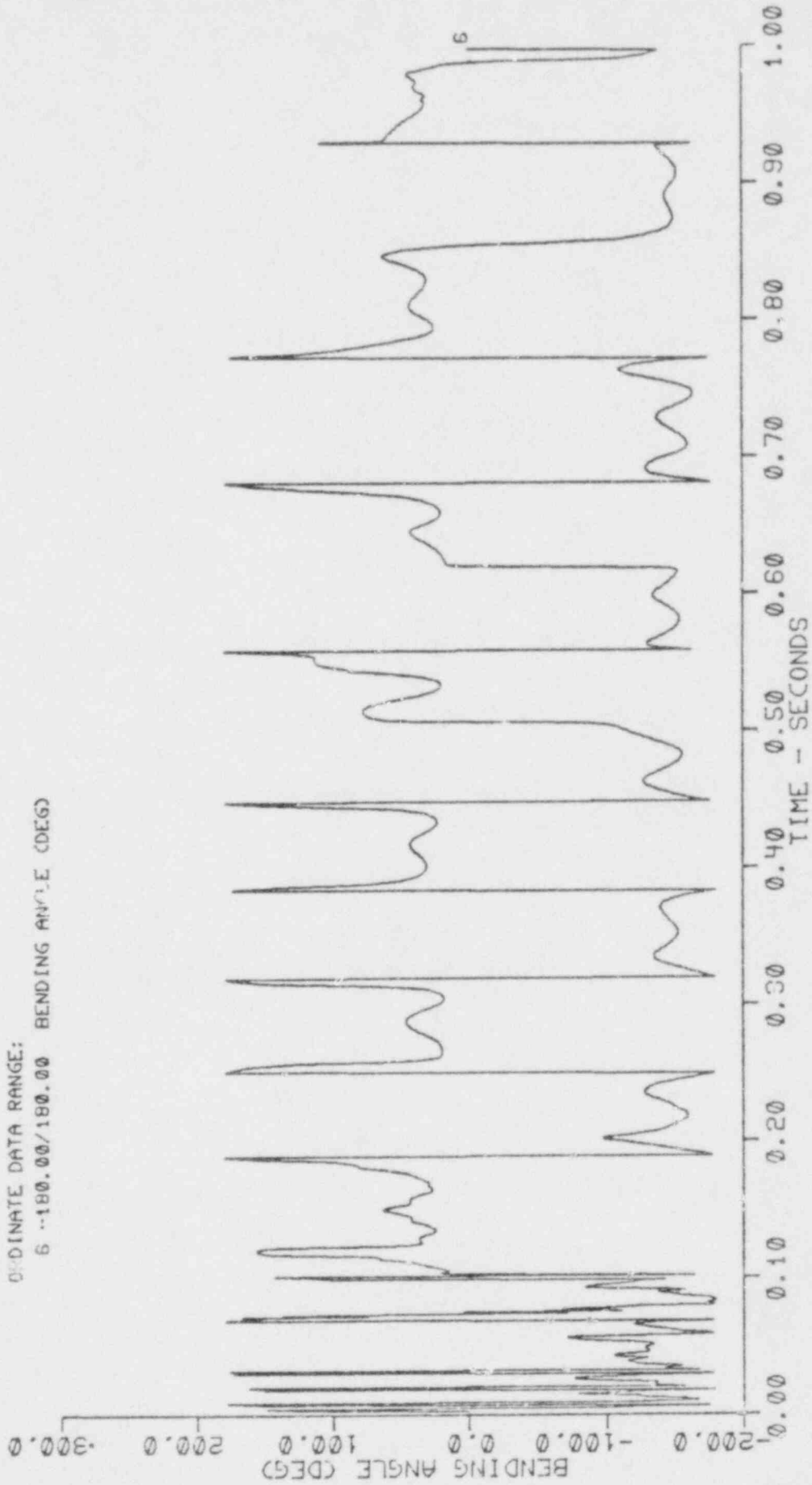
SP4AHDRSRV350

V60.4

RUN:

TEST: COORDINATE DATA RANGE:

6 --180.00/180.00 BENDING ANGLE (DEG)



CHANNEL 6 RK2212 GRAD



FIGURE 5.28

V60.4

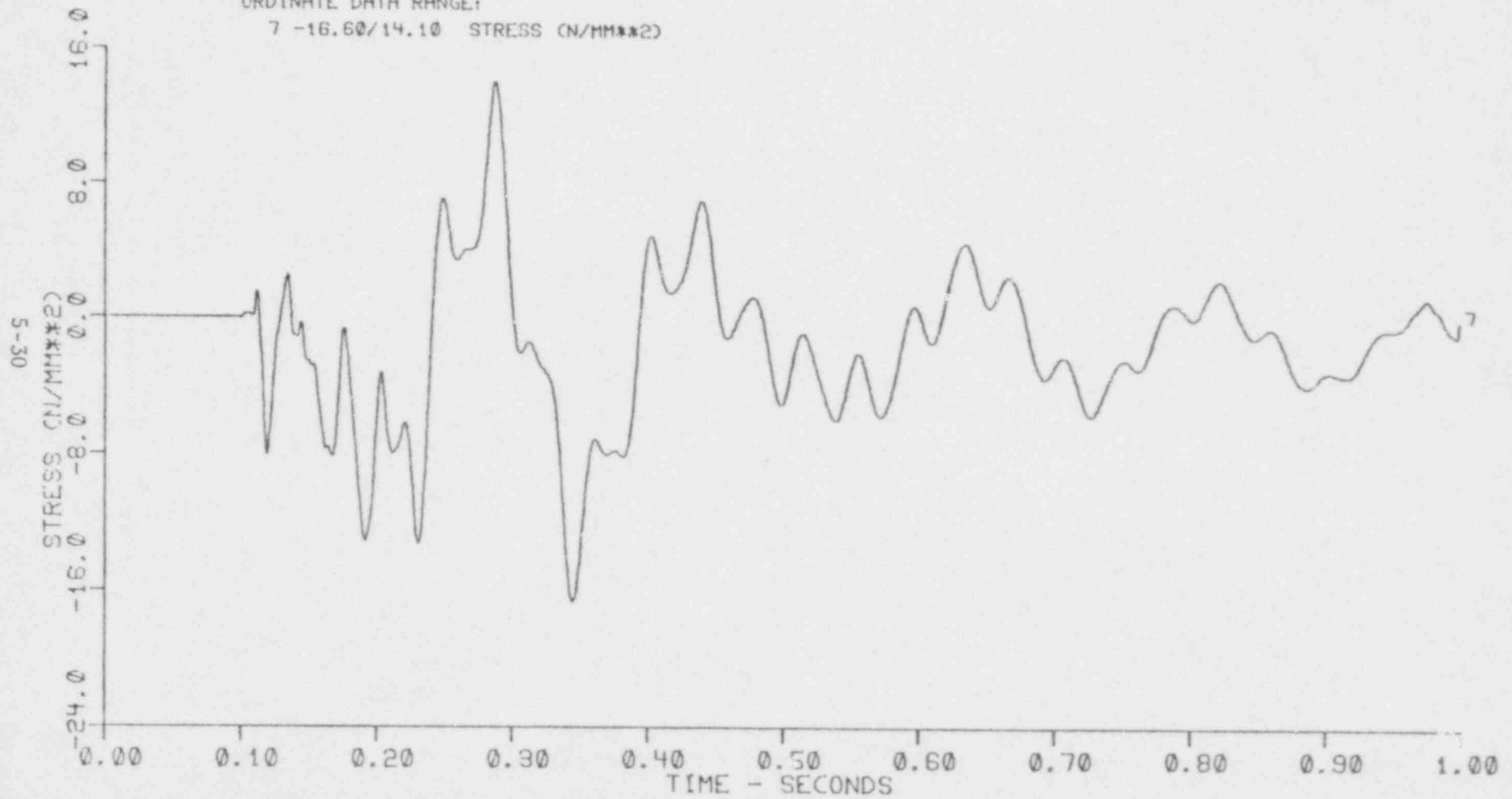
SP4AHDRSRV350

TEST:

PLIN:

ORDINATE DATA RANGE:

7 -16.60/14.10 STRESS (N/MM\*\*2)



CHANNEL 7

RK3012

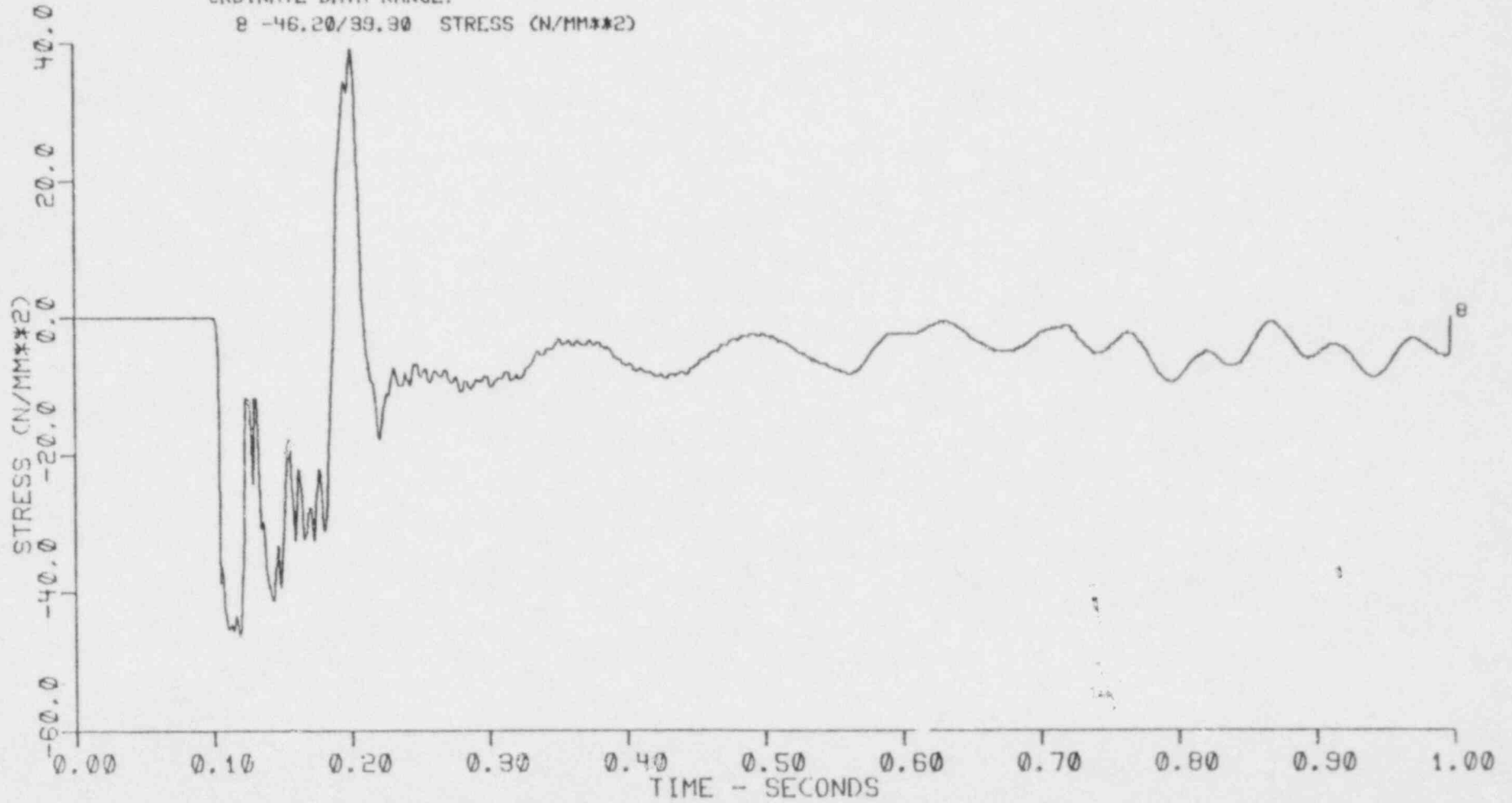
N/MM2

FIGURE 5.29  
V60.4

SP4AHDRSRV350

TEST:                    RUN:  
ORDINATE DATA RANGE:  
8 -46.20/39.30 STRESS (N/MM\*\*2)

IS-S



CHANNEL 8

RK4112

N/MM2

FIGURE 5.30

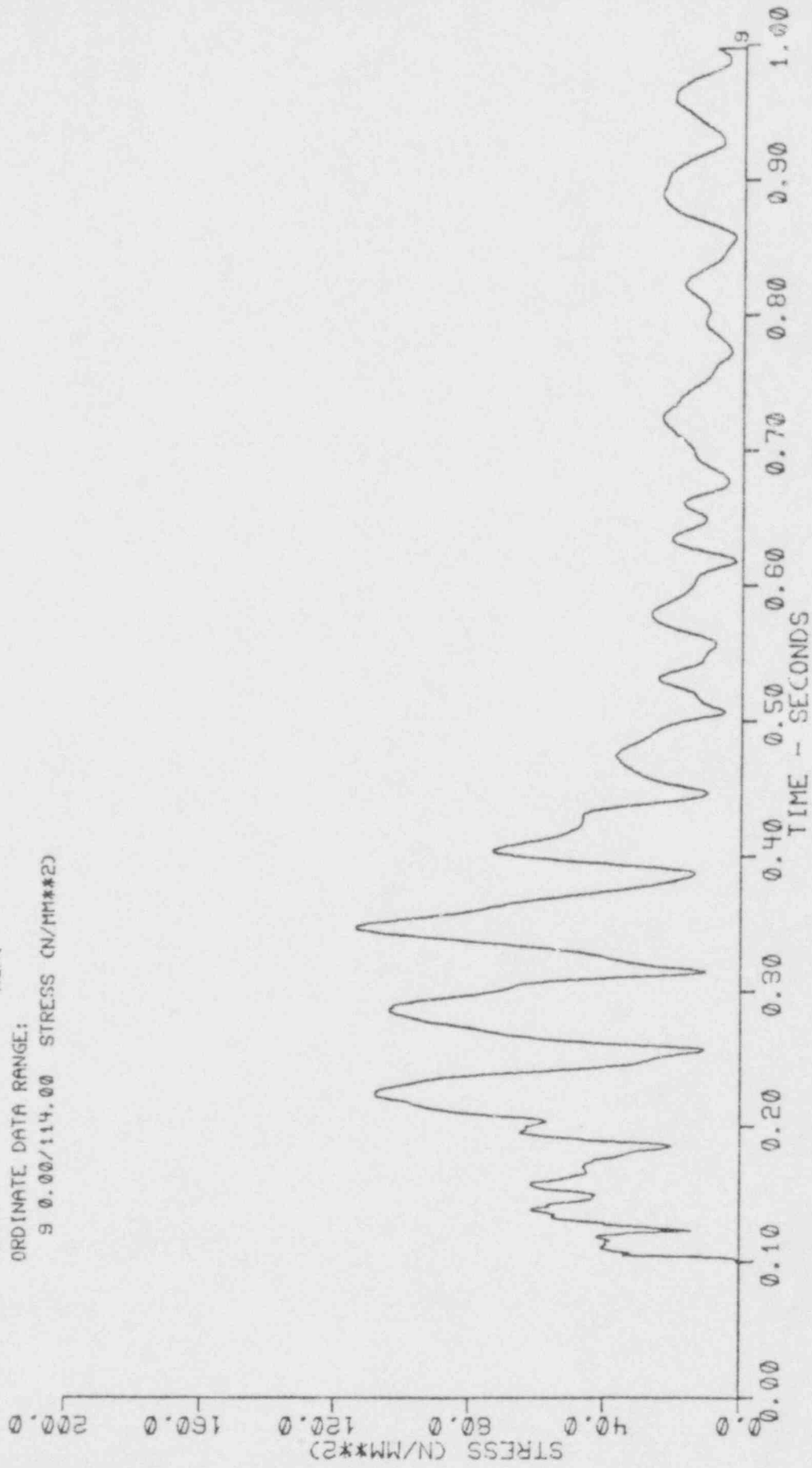
V60.4

SP4AHD03RV350

TEST: RUM:

ORDINATE DATA RANGE:

9 0.00/114.00 STRESS (N/MM\*\*2)



CHANNEL 9 RKS012 N/MM2

FIGURE 5.31

V60.4

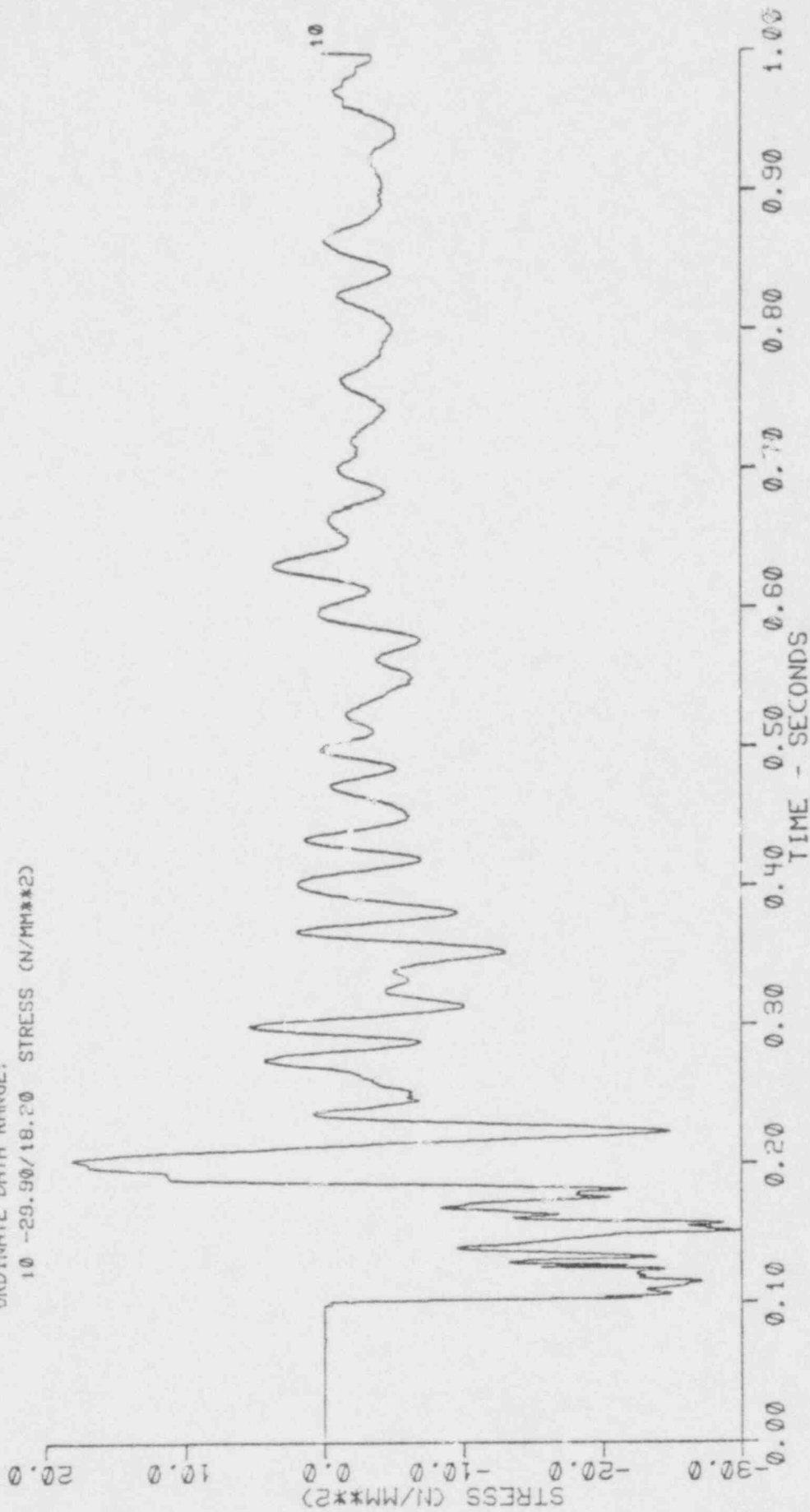
SP4AHDRSRV350

TEST:

RUN:

ORDINATE DATA RANGE:

10 -29.90/18.20 STRESS (N/MM\*\*2)



CHANNEL 10 RI1013 N/MM2

FIGURE 5.32

(Problem with file information for RK 2013)

FIGURE 5.33

V60.4

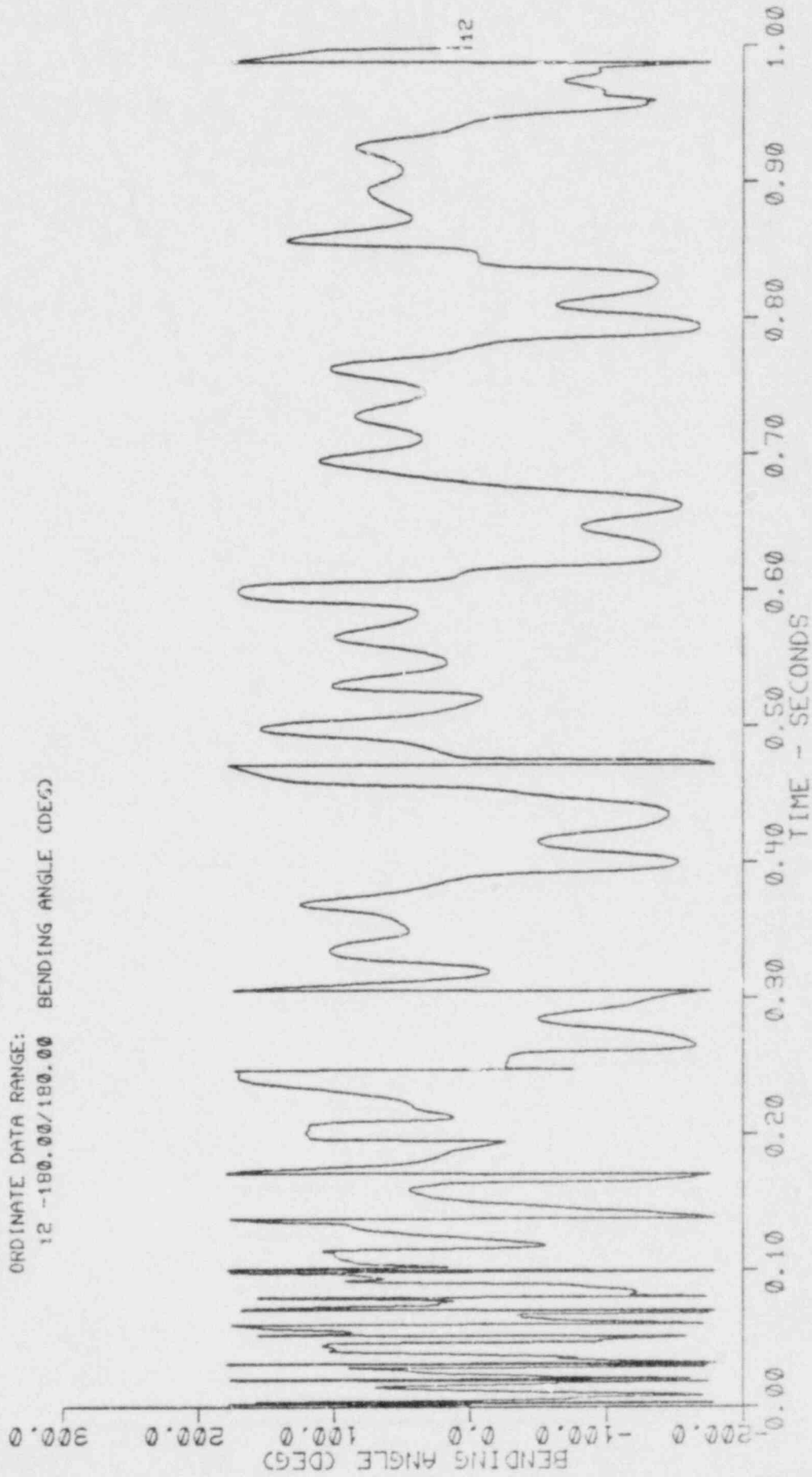
SP4AHDRSRV350

RUN:

TEST:

ORDINATE DATA RANGE:

12 -180.00/180.00 BENDING ANGLE (DEG)



CHANNEL 12 RK2219 GRAD

FIGURE 5.34

V60.4

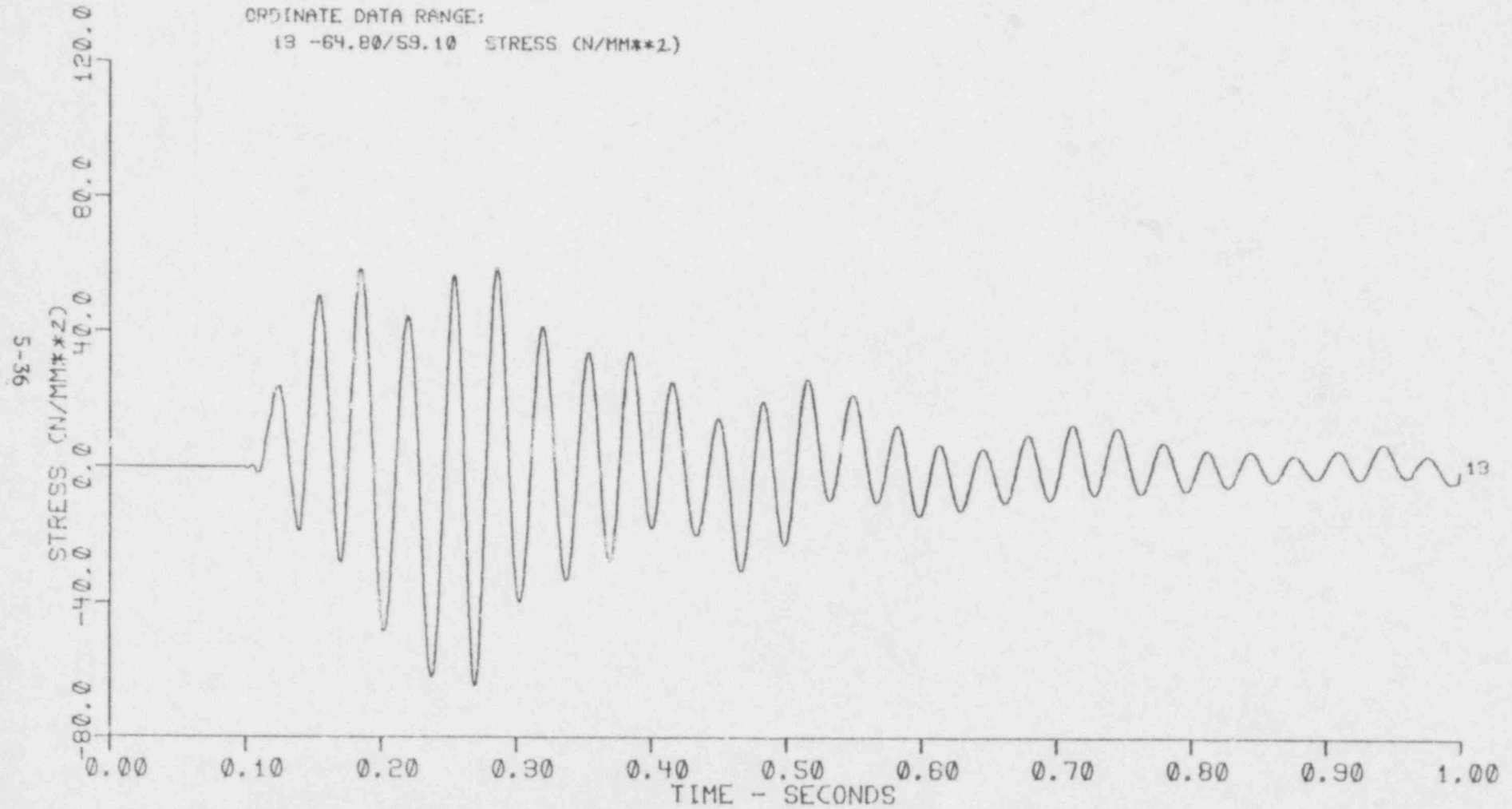
SP4AHDRSRV350

TEST:

RUN:

ORDINATE DATA RANGE:

13 -64.00/59.10 STRESS (N/MM\*\*2)



CHANNEL 13

RK3013

N/MM2

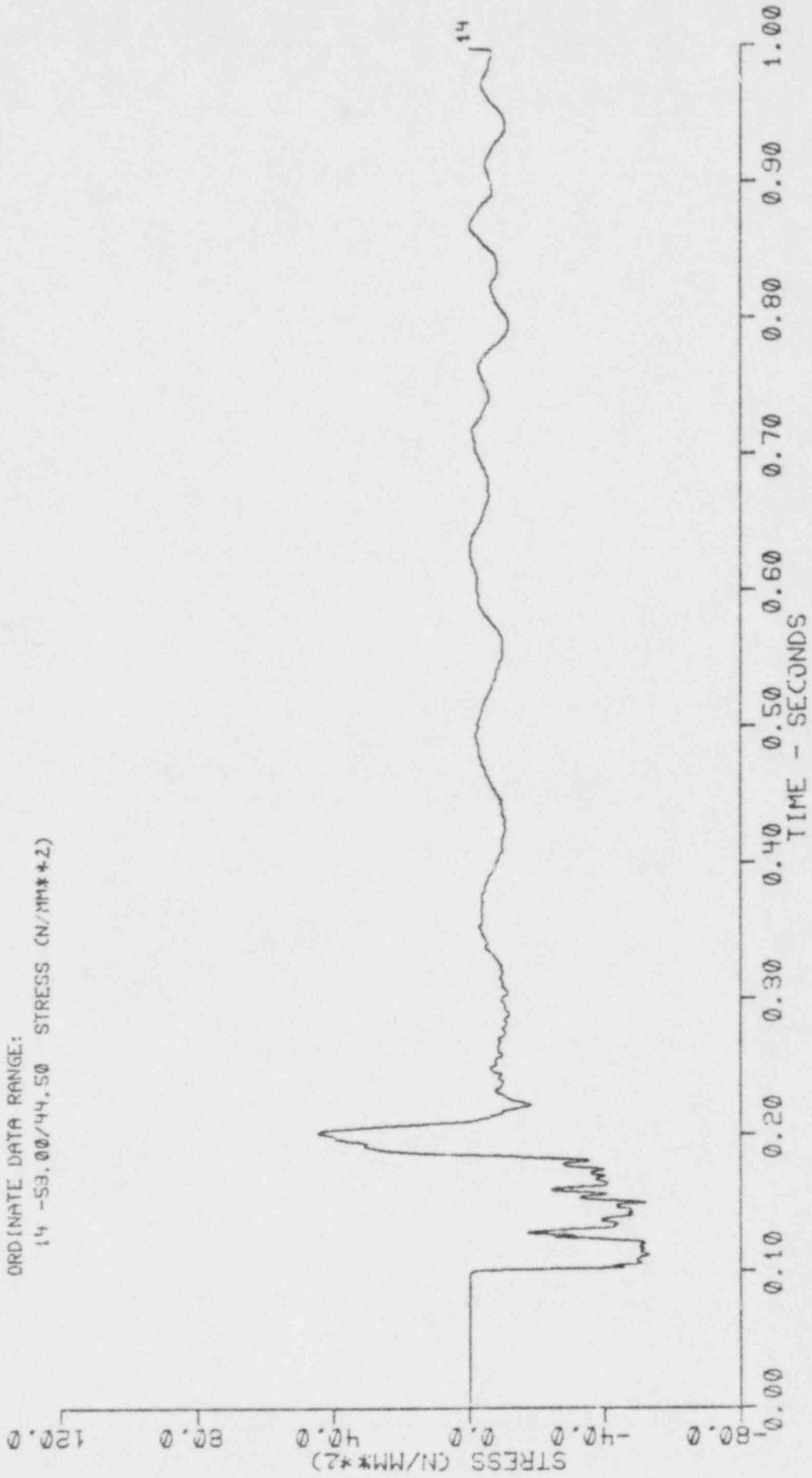
FIGURE 5.35

V60.4 SP4AHDRSRV350

TEST: RUN:

ORD (NATE DATA RANGE:

14 -53.00/44.50 STRESS (N/MM\*\*2)



CHANNEL 14 RK4113 N/MM2



FIGURE 5.36  
V60.4

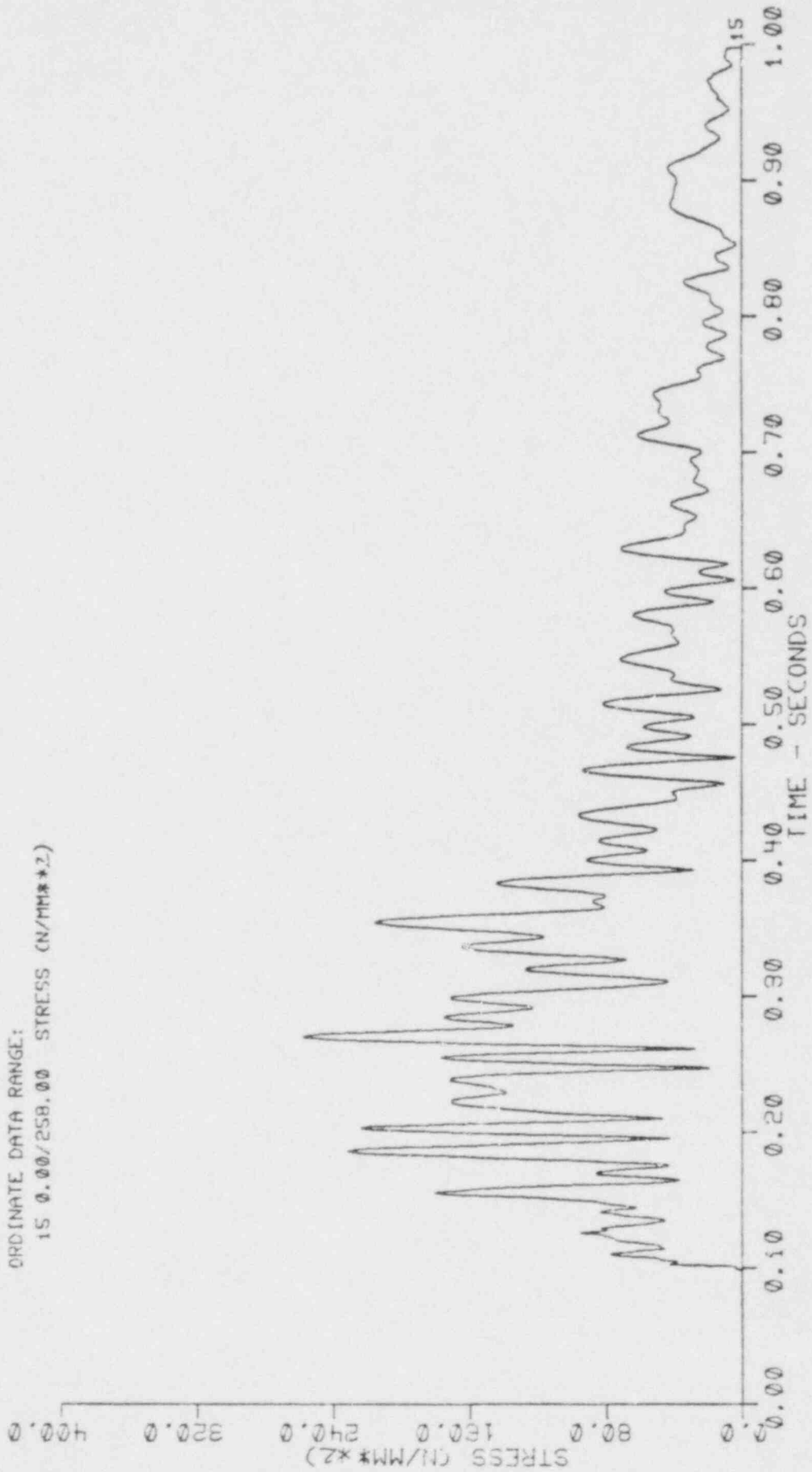
SP4HDRSRV350

RUN:

TEST:

ORDINATE DATA RANGE:

15 0.00/258.00 STRESS (N/MM\*\*2)



CHANNEL 15 RK5013 N/MHC

FIGURE 5.37

V60.4

SP4AHDRSRV350

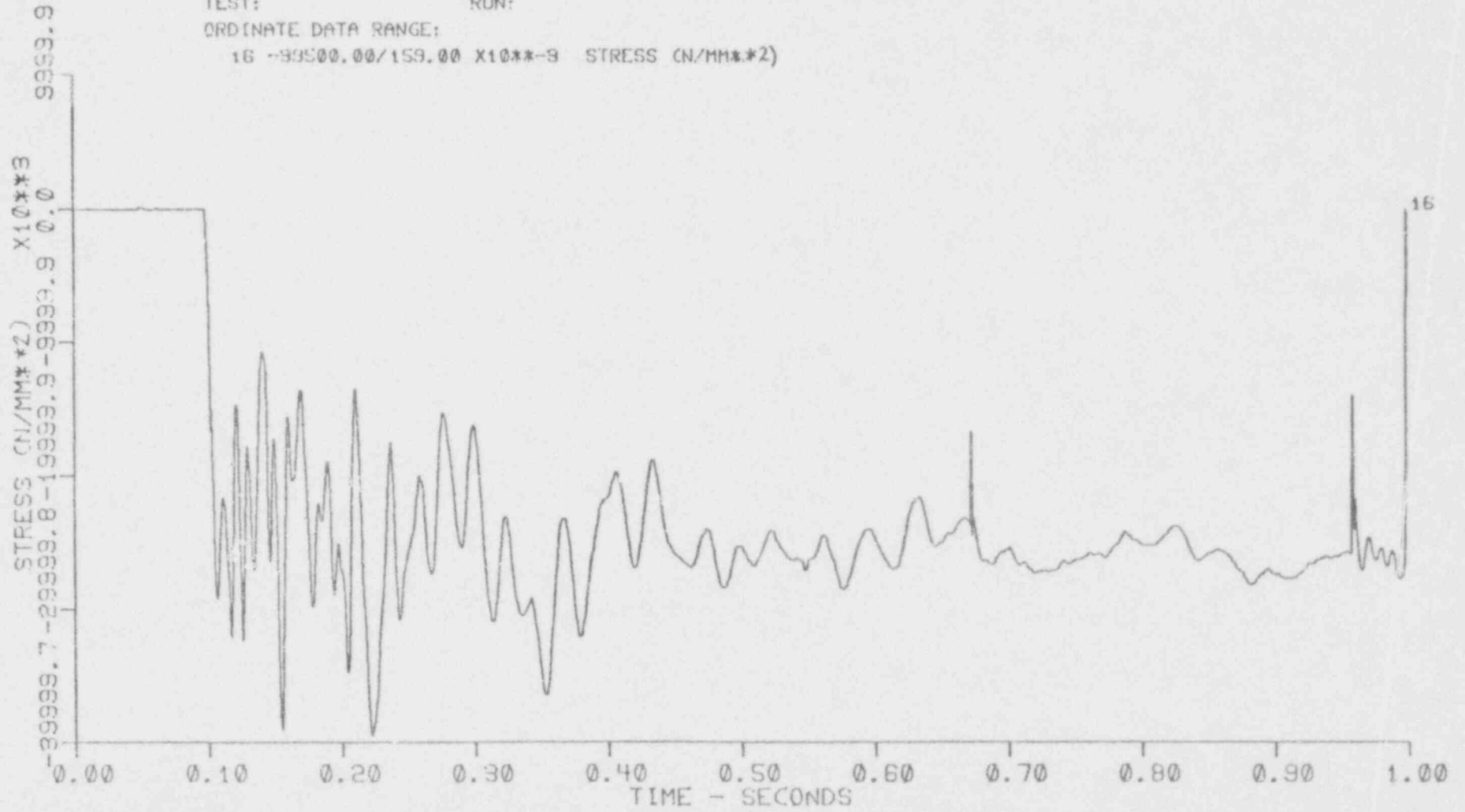
TEST:

RUN:

ORDINATE DATA RANGE:

16 -99999.99/159.99 X10\*\*3 STRESS (N/MM\*\*2)

5-39



CHANNEL 16

RK1014

N/MM2

FIGURE 5.38

(Problem with file information for RK 2014)

FIGURE 5.39

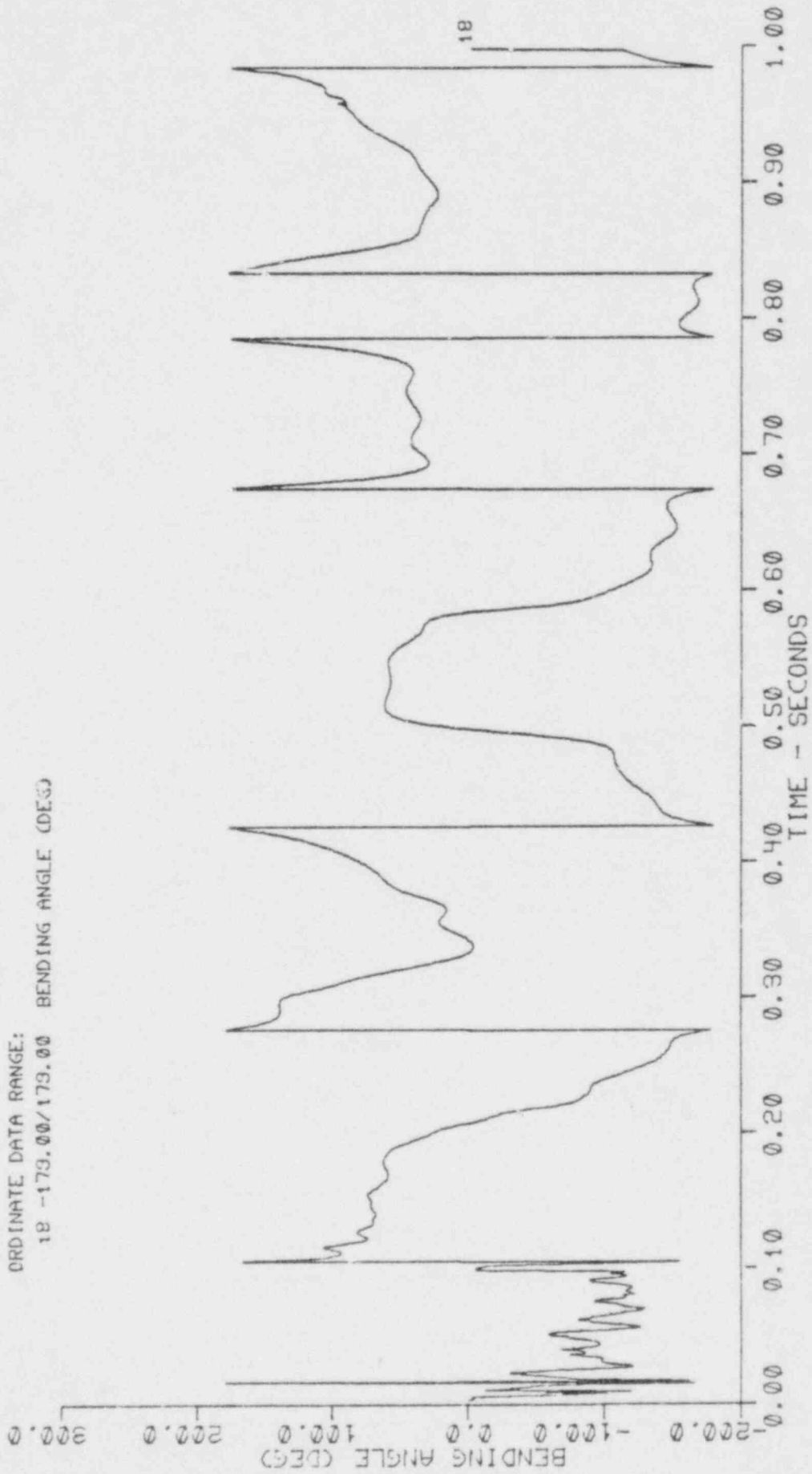
V60.4

SP4AHDRSRV350

TEST:

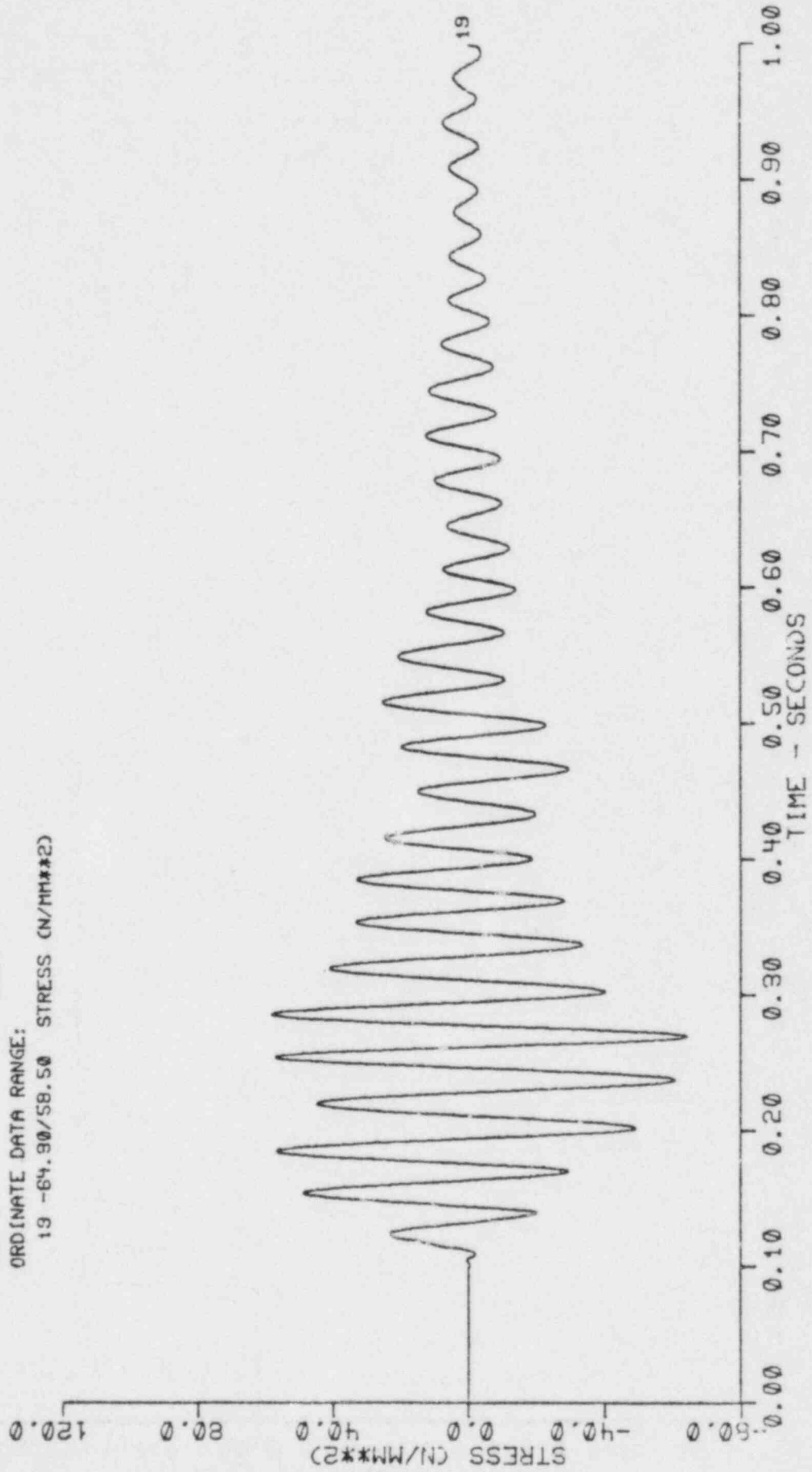
ORDINATE DATA RANGE:

18 -179.00/179.00 BENDING ANGLE (DEG)



CHANNEL 18 RK2214 GRAD

FIGURE 5.40  
V60.4  
SP4AHDRSRV350  
TEST: RUN:  
ORDINATE DATA RANGE:  
19 -64.90/58.50 STRESS (N/MM\*\*2)



CHANNEL 19 RK3014 N/MM2

FIGURE 5.41

VS0.4

SP4AHDRSRV350

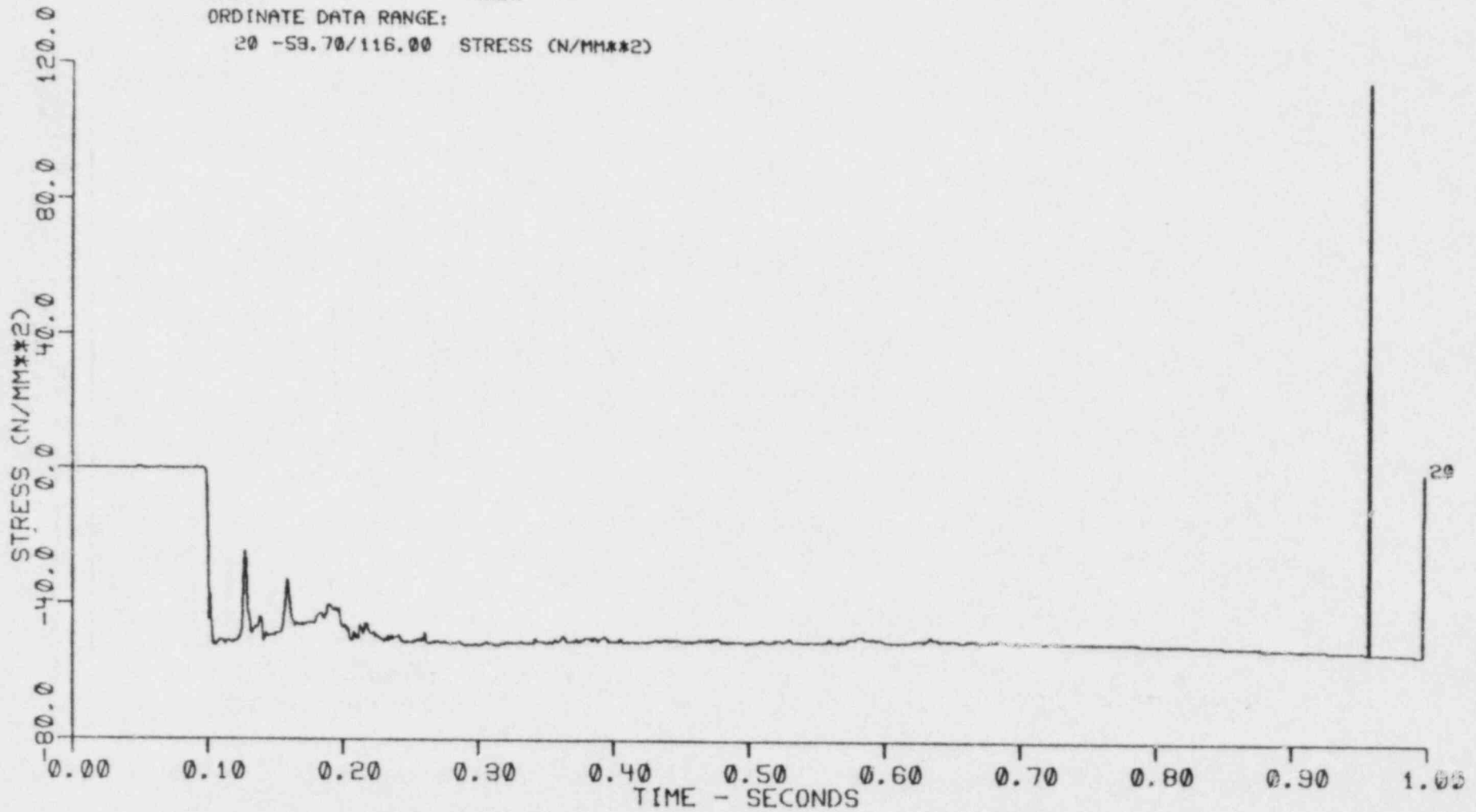
TEST:

RUN:

ORDINATE DATA RANGE:

20 -53.70/116.00 STRESS (N/MM\*\*2)

5-43



CHANNEL 20

RK4114

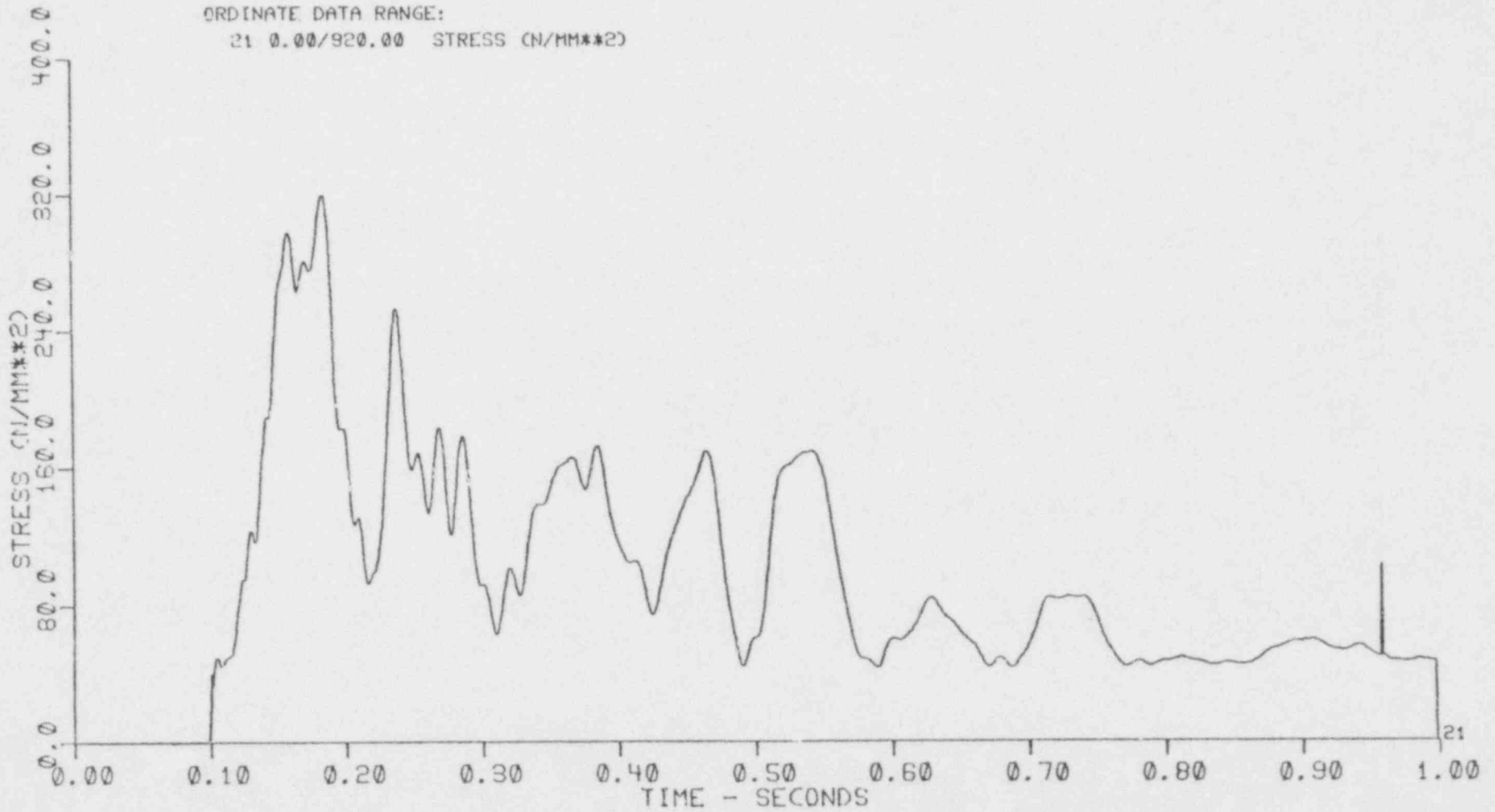
N/MM2

FIGURE 5.42  
V60.4

SP4AHDRSRV350

TEST:                    RUN:  
ORDINATE DATA RANGE:  
21 0.00/920.00 STRESS (N/MM\*\*2)

5-44



CHANNEL 21

RK5014

N/mm<sup>2</sup>

TABLE 5.1

EXTREME VALUES OF STRUCTURAL RESPONSE  
DISPLACEMENTS AND ACCELERATIONS

<u>Response Number</u>	<u>ANCO Node</u>	<u>Response Direction</u>	<u>Minimum/Maximum Value</u>
RS2201	14	y	-49/44 (mm)
RS2202	14	x	-20/20 (mm)
RS2203	14	z	-47/43 (mm)
RS2204	22	y	-87/67 (mm)
RS2205	22	z	-39/43 (mm)
RS2206	22	x	-50/41 (mm)
SS4004	43	z	-18/21 (mm)
SS4005	43	x	-1/2 (mm)
SS4006	43	y	-15/18 (mm)
SS4001	42	x	-38/7 (g's)
SS4002	42	y	-23/26 (g's)
SS4003	42	z	-19/18 (g's)



## 6.0 COMPARISON OF STRUCTURAL RESPONSE TEST DATA AND ANCO SIMULATION RESULTS

A brief comparison is made, herein, between some of the test data and the results of the simulation performed by ANCO Engineers, Inc. The quantities that have been compared are: (1) eigenvalues of the pipe system and; (2) maximum and minimum displacements of the pipe during the dynamic event. The test data was taken from the report "Ergebnisbericht, Blowdown-Versuch NR. V60.4.1, am 5.12.80, Versuchsgruppe SRV 350," Februar 1981, Kernforschungszentrum Karlsruhe, Projekt HDR.

The experimentally determined and predicted eigenfrequencies are given in Table 6.1. There was a problem in comparing the two sets of values. Only two experimental frequencies were given below 25 Hz, whereas, there were three predicted frequencies. This difference was partially resolved through the following discussion. There is a large relative difference between the third and fourth theoretical eigenfrequencies. There is also a large difference between the second and fourth experimental eigenfrequencies (as defined in Table 6.1). Because of this, together with the fact that the values of the frequencies (experimental and predicted), as given for the fourth and fifth modes, are close to each other, it is believed that the experimental frequencies given in Table 6.1 for the fourth and fifth modes probably correspond to the fourth and fifth theoretical frequencies.

In comparing the eigenfrequencies below 25 Hz, there are essentially two possibilities. First, that there exist only two experimental frequencies below 25 Hz; or second, that there exist more experimental frequencies below 25 Hz, one of which was not detected during data analysis. Regardless of what the situation is, it is most logical that the first experimental frequency corresponds to the first theoretical frequency. If it did not, the first experimental frequency would correspond to the second theoretical frequency giving the theoretical frequency a relative error of 73 percent. With good agreement between theory and experiment for the fourth and fifth eigenfrequencies, and with the physical system being

TABLE 6.1  
COMPARISON OF EXPERIMENTAL AND  
THEORETICAL EIGENFREQUENCIES

<u>Mode</u>	<u>Experimental Eigenfrequency (Hz)</u>	<u>Predicted Eigenfrequency (Hz)</u>	<u>Relative Difference (%)</u>
1	4.95	5.46	10.3
2	7.75	8.54	10.2
3	*	9.43	*
4	25.50	26.18	2.7
5	29.30	30.55	4.3

---

\*Note: On the assumption that the first two experimental eigenfrequencies correspond to the first two theoretical eigenfrequencies, either there is not an experimental mode that corresponds to the third theoretical mode (9.43 Hz eigenfrequency mode) or the third experimental mode was not observed during analysis of the experimental data by German investigators.

essentially linear for non-plastic deformation and hence being very amenable to modeling as a linear system, an error of 73 percent for the second frequency is improbable. Overall, there seems to be excellent agreement between the two sets of eigenfrequencies.

The maximum and minimum displacements the pipe experienced during the dynamic test (DSP4) are compared to the predicted results in Table 6.2. There is a substantial difference between experiment and theory. Some of the large difference occurs where the displacement is small (i.e., SS4005) and, hence, does not have a great deal of meaning. Some of the large difference occurs where the displacement is large (i.e., RS2204). This difference is of concern. Some of the small to moderate differences occur where the displacements range from being small to large. Overall, the comparison of theory to experiment seems to be fair, with the predicted values bounding the experimental values.

TABLE 6.2  
COMPARISON OF MAXIMUM AND MINIMUM DISPLACEMENTS

<u>Transducer</u>	<u>Minimum/Maximum Value (mm)</u>		<u>Relative Difference (%)</u>
	<u>Experimental</u>	<u>Predicted</u>	
RSS2202	-7/9	-20/20	186/122
RS2201	-35/37	-49/44	40/19
RS2203	-22/18	-47/43	114/139
RS2206	-/-	-50/41	-/-
RS2204	-52/62	-87/67	67/8
RS2205	(-30/30)*	-59/43	30/43
SS4005	-7/8	-1/2	85/75
SS4004	-23/27	-18/21	22/22
SS4006	-13/17	-15/18	15/6

\*Note: Displacement transducer may have been damaged during the test.

## 7.0 COMMENTS

Even though the DSP4a theoretical simulation was to have been as close to reality as possible it was necessary to make certain simplifying assumptions due to reasonable resource constraints and limited fluid dynamic response data from DSP4. Following is a list of these assumptions:

### Structural Assumptions

- A linear structural simulation would give meaningful results.
- Each end of the pipe line was fixed.
- The pipe could be represented with pipe elements (essentially straight and curved beam elements); no ovaling of the pipe elbows would occur to any significant degree.
- The damping effects could be represented with proportional damping and the damping was between 3.0% and 4.5% of critical.
- The fluid-structure problem could be decoupled and still give reasonably correct results.

### Fluid Assumptions

- The discretization of the space inside the URL pipe into the six defined control volumes (Figure 4.2) was fine enough to generate a satisfactory load distribution on the pipe.
- The net rate of momentum efflux from a control volume and the rate of change of momentum within a control volume can be neglected as compared to the fluid pressure surface force of the volume.
- Body forces are negligible.
- Choosing the geometric center of a control volume as the center of pressure will not seriously affect the results.
- No significant concentrated moments on the pipe would be generated by the control volumes.

The structural dynamic simulation was performed using the above assumptions. The response information required by DSP4a was generated. As far as can be determined by inspection of the simulation results, it appears that there are no serious problems with the solution approach or the solution.

APPENDIX A

URL BLOWDOWN FINITE ELEMENT MODEL  
(EASE2 INPUT LISTING)

APPENDIX A

URL BLOWDOWN FINITE ELEMENT MODEL

(EASE2 Input Listing)

Following is a listing of the data which defines the EASE2 model of the URL blowdown pipe system.

# EASE2 INPUT

## MASTER CONTROL PARAMETERS

HDR / URL BLOWDOWN MODEL

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MASTER ECHO PRINT CONTROL EQ. 9, SUPPKESS ECHO PRINT IN ALL DATA SECTIONS	= ( 1 )	SPS (CC 15)
SOLUTION MODE CONTROL EQ. 1, STATIC SOLUTION EQ. 2, EIGENVALUE SOLUTION ONLY EQ. 3, TIME-HISTORY ANALYSIS USING MODE SUPERPOSITION EQ. 4, TIME HISTORY ANALYSIS USING DIRECT INTEGRATION EQ. 5, RESPONSE SPECTRUM ANALYSIS	= ( 4 )	IMODE (CC 15-20)
A-3 STIFFNESS MATRIX RESTART CONTROL EQ. 1, PROGRAM EXPECTS TO READ THE DECOMPOSED STIFFNESS MATRIX FRGM DISK FILE TAPE12	= ( 0 )	IRSTR1 (CC 24)
EIGENVALUES/EIGENVECTORS RESTART CONTROL EQ. 1, PROGRAM EXPECTS TO READ EIGENVALUES AND EIGENVECTORS FROM DISK FILE TAPE13	= ( 0 )	IRSTR2 (CC 25)
NODE RESEQUENCING CONTROL EQ. 0, INTERNAL MODE NUMBERS ASSIGNED IN ASCENDING NODE ORDER EQ. 1, INTERNAL NODE NUMBERS ASSIGNED SEQUENTIALLY AS NODES ARE ENCOUNTERED DURING INPUT EQ. 2, INTERNAL NODE NUMBERS ASSIGNED TO PRODUCE A REDUCED BANDWIDTH USING *GPS* ALGORITHM BLANK, DEFAULT SET TO 2	= ( 2 )	IBAND (CC 25-30)
MAXIMUM CORE FOR EASE2 EXECUTION BLANK, DEFAULT SET TO 250000 LT. 1000008, RESET TO 1000008	= (250000)	ICORE (CC 35-40)



COORDINATE SYSTEM DATA

HDX / URL BLOWDOWN MODEL

COORDINATE SYSTEM NUMBER	SYSTEM TYPE	COORDINATES OF THE ORIGIN			PROJECTIONS OF CARTESIAN (X*) OR CYLINDRICAL (Z*) GLOBAL X GLOBAL Y GLOBAL Z			PROJECTIONS OF CARTESIAN (Y*) OR CYLINDRICAL (R*) GLOBAL X GLOBAL Y GLOBAL Z		
2	0	-2.421	17.300	.650	*.259	0.000	*.966	*.966	0.000	0.000
3	0	-2.170	17.300	1.253	*.500	0.000	*.866	*.866	0.000	-*.500
4	0	-1.832	17.300	2.373	-.500	0.000	*.866	*.866	0.000	*.500
5	0	-2.846	19.755	4.130	0.000	-1.000	0.000	*.866	0.000	*.500
6	0	-3.374	11.500	3.325	-.866	0.000	-.500	-.500	0.000	*.966
7	0	-2.506	17.300	-.520	0.000	0.000	1.000	1.000	0.000	0.000

INPUT NODE DATA

HDR / URL BLOWDOWN MODEL

NODE NO	SYSTEM / TYPE	COORDINATES OF THE FIRST NODE			THICKNESS	G E N E R A T I O N		COORDINATES OF THE LAST NODE			NODE COUNT
		X1-ORD	X2-ORD	X3-ORD		PERCENT CHANGE	2ND-LEVEL CODE	X1-ORD	X2-ORD	X3-ORD	
2	0/	0.000	17.300	0.000	0.00						1
4	0/	-1.379	17.300	-0.796	0.00						2
6	0/	-1.703	17.300	-0.984	0.00						3
8	0/	-2.506	17.300	-0.520	0.00						4
10	0/	-2.506	17.300	0.000	0.00						5
11	7/	1.170	-0.085	0.000	0.00						6
12	0/	-2.170	17.300	1.253	0.00						7
14	0/	-1.832	17.300	1.838	0.00						8
16	0/	-1.832	17.300	2.373	0.00						9
18	0/	-2.206	17.300	3.020	0.00						10
20	0/	-2.579	17.300	3.667	0.00						11
22	0/	-2.846	16.765	4.130	0.00						12
24	0/	-2.846	15.834	4.130	0.00						13
26	0/	-2.846	14.903	4.130	0.00						14
28	0/	-2.846	13.971	4.130	0.00						15
30	0/	-2.846	13.040	4.130	0.00						16
32	0/	-2.846	12.550	4.130	0.00						17
34	0/	-2.846	12.209	4.130	0.00						18
36	0/	-2.846	12.109	4.130	0.00						19
37	6/	-0.431	0.000	-0.179	0.00						20
38	0/	-3.374	11.500	3.625	0.00						21
40	0/	-3.359	11.500	3.545	0.00						22
42	0/	-4.275	11.500	3.305	0.00						23
43	0/	-4.790	11.500	3.010	0.00						24
44	0/	-5.314	11.500	2.705	0.00						25
46	0/	-5.747	11.500	2.455	0.00						26
48	0/	-5.920	11.500	2.358	0.00						27
49	6/	3.250	-0.084	0.000	0.00						28
50	0/	-6.230	11.500	1.830	0.00						29
52	0/	-6.287	11.500	1.308	0.00						30

J K J E K E D N O D E O A T A (GLOBAL CARTESIAN COORDINATES)

HCR / URL BLOWDOWN MODEL

NODE NUMBER	GLOBAL X-ORD	GLOBAL Y-ORD	GLOBAL Z-ORD	THICKNESS	NODE NUMBER	GLOBAL X-ORD	GLOBAL Y-ORD	GLOBAL Z-ORD	THICKNESS
2	0.0000	17.3000	0.0000	0.00					
4	-1.3790	17.3000	-0.7960	0.00					
6	-1.7030	17.3000	-0.9840	0.00					
8	-2.5060	17.3000	-0.5200	0.00					
10	-2.5060	17.3000	0.0000	0.00					
11	-2.4210	17.3000	0.6500	0.00					
12	-2.1700	17.3000	1.2530	0.00					
14	-1.8320	17.3000	1.8380	0.00					
16	-1.8320	17.3000	2.3730	0.00					
18	-2.2060	17.3000	3.0200	0.00					
20	-2.5790	17.3000	3.6670	0.00					
22	-2.8460	16.7650	4.1300	0.00					
24	-2.8460	16.8340	4.1300	0.00					
26	-2.8460	15.9030	4.1300	0.00					
28	-2.8460	13.9710	4.1300	0.00					
30	-2.8460	13.0400	4.1300	0.00					
32	-2.8460	12.5000	4.1300	0.00					
34	-2.8460	12.2090	4.1300	0.00					
36	-2.8460	12.1090	4.1300	0.00					
37	-3.0007	11.6790	4.0405	0.00					
38	-3.3740	11.5000	3.8250	0.00					
40	-3.8590	11.5000	3.5450	0.00					
42	-4.2750	11.5000	3.3050	0.00					
43	-4.7700	11.5000	3.0100	0.00					
44	-5.3140	11.5000	2.7050	0.00					
46	-5.7470	11.5000	2.4550	0.00					
48	-5.9200	11.5000	2.3580	0.00					
49	-6.1466	11.5000	2.1272	0.00					
50	-6.2300	11.5000	1.8300	0.00					
52	-6.2870	11.5000	1.3080	0.00					

RESTRAINED NODES

HDR / URL BLOWDOWN MODEL

GENERAL RESTRAINT CODE APPLIED TO ALL NODES (000000) SGC00E (CC 25-30)

NODE NUMBER	BOUNDARY CODES	/TRANSLATIONAL COMPONENT VALUES/ (X OR R) (Y OR S) (Z OR T)	/ ROTATIONAL COMPONENT (X OR R) (Y OR S) (Z OR T)	VALUES / (Z OR T)	GENERATION NUMBER	INCREMENT	NODE COUNT
2	DDD DDD	0* 0* 0*	0* 0* 0*	0*	0	1	1
4	DDD DDD	0* 0* 0*	0* 0* 0*	0*	0	1	2
5	RRR RRR	0* 0* 0*	0* 0* 0*	0*	0	1	3
52	RRR RRR	0* 0* 0*	0* 0* 0*	0*	0	1	4

MATERIAL PROPERTIES

HDR / URL BLOWDOWN MODEL

MATERIAL NUMBER	MATERIAL DESCRIPTION	TEMPERATURE	WEIGHT DENSITY	ELASTIC MODULUS	POISSON'S RATIO	EXPANSION COEFFICIENT
1	CPIPE	0.	.1037E+06	.1987E+12	.2900E+00	0.
2	SPIPE	0.	.1123E+06	.1987E+12	.2900E+00	0.
3	SPIPE	0.	.1262E+06	.1987E+12	.2900E+00	0.
4	CPIPE	0.	.1091E+06	.1987E+12	.2900E+00	0.
5	VALVE	0.	.2118E+05	.1987E+12	.2900E+00	0.
6	SPIPE	0.	.9610E+05	.1987E+12	.2900E+00	0.
7	SPIPE	0.	.1199E+06	.9990E+12	.2900E+00	0.

PIPE SECTION PROPERTIES

HDR / URL BLOWDOWN MODEL

SCALE FACTOR - DIVIDE DIAMETER AND  
WALL THICKNESS ENTRIES BY (FLU)

( 0.00003 ) FLU (CC 21-30)

SECTION NUMBER	SHEAR EFFECTS	OUTSIDE DIAMETER	WALL THICKNESS	SHAPE FACTOR FOR SHEAR	SECTION DESCRIPTION
1	YES	.41000	.02500	1.99440	
2	YES	.39860	.01930	1.99656	
3	YES	.40500	.01400	1.99829	
4	YES	.40600	.02500	1.99428	
5	YES	.40600	.01750	1.99730	
6	YES	.45500	.64180	1.98649	
7	YES	1.20000	.60000	1.33333	
8	YES	.50800	.04000	1.99033	

P I P E E L E M E N T S

HUR / URL BLOWDOWN MODEL

PIPE NUMBER AND TYPE	NODE 1	NODE 2	MATL ID	TUBE SECT ID	END RELEASE ID	GEN ER AT I O N - /1ST LEVEL/ /2ND LEVEL/ /3RD PT /SPEC /SYST	3RD PT /SPEC /SYST	A D D I T I O N A L X3-ORD /3END /ADJJS	B E N D Y3-ORD /FACTOR (P,K)	O A T A Z3-ORD /TOLER- ANCE	TEMPE- RA TURE	ELEMENT LENGTH	WIDTH
18	6	8	1	1	0	0	1	0.00 .535 -1.97	.54 ***** 17.30	0.00 .10000 -.52)	0.0	.93	12
2	8	10	2	2	0	0	1				0.0	.52	12
38	10	11	1	2	0	0	1	0.00 2.506 .00	2.51 ***** 17.30	0.00 .10000 -.00)	0.0	.06	12
4	12	14	2	2	0	0	1				0.0	.63	12
53	14	16	1	1	0	0	1	0.00 .535 -2.30	-.54 ***** 17.30	0.00 .10000 2.11)	0.0	.54	12
6	16	18	2	2	0	0	1				0.0	.75	12
7	18	20	2	2	0	0	1				0.0	.75	12
83	20	22	1	1	0	0	1	1.49 .535 -2.22	0.00 ***** 16.77	-.54 .10000 3.67)	0.0	.76	12
9	22	24	2	2	0	0	1				0.0	.93	12
10	24	26	2	3	0	0	1				0.0	.93	12
11	26	28	2	2	0	0	1				0.0	.93	12
12	28	30	2	2	0	0	1				0.0	.93	12
13	30	32	2	2	0	0	1				0.0	.49	12
14	32	34	2	2	0	0	1				0.0	.34	12
15	34	36	3	1	0	0	1				0.0	.10	12
168	36	37	4	4	0	0	1	4.66 .610 -3.37	-.61 ***** 12.11	0.00 .10000 3.63)	0.0	.47	12
17	38	40	3	2	0	0	1				0.0	.56	12
18	40	42	6	6	0	0	1				0.0	.48	12
19	42	43	5	7	0	0	1				0.0	.57	12
20	44	46	6	6	0	0	1				0.0	.50	12
21	46	48	3	5	0	0	1				0.0	.20	12
228	48	49	4	4	0	0	1	2.74 .610 -5.62	-.61 ***** 11.50	0.00 .10000 1.83)	0.0	.32	12
23	50	51	7	8	0	0	1				0.0	.53	12
248	11	12	1	2	0	0	1	0.00 2.506 .00	2.51 ***** 17.30	0.00 .10000 -.60)	0.0	.65	12
258	37	38	4	4	0	0	1	4.66 .610 -3.37	-.61 ***** 12.11	0.00 .10000 3.63)	0.0	.47	12
26	43	44	5	7	0	0	1				0.0	.61	12
278	49	50	4	4	0	0	1	2.94 .610 -5.62	-.61 ***** 11.50	0.00 .10000 1.83)	0.0	.31	12

CONCENTRATED FORCES (LOAD CASE 1)

HDR / URL BLOWDOWN MODEL

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LOAD CASE 1 --

NODE NUMBER	SPEC SYST	SKEW NODE	REST- RAINT	CONS- TRANT	REPEAT NODE	(X,R OR X*) FORCE	(Y,S OR Y*) FORCE	(Z,T OR Z*) FORCE	(X,R OR X*) MOMENT	(Y,S OR Y*) MOMENT	(Z,T OR Z*) MOMENT	*GENERATION* NUMBER	INC COUNT
49	6					.10000E+01	0.	0.	0.	0.	0.	0	1 1



C O N C E N T R A T E D F O R C E S (LOAD CASE 2)

HDR / UAL BLOWDOWN MODEL

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LOAD CASE 2 --

NODE	SPEC	SKEM	REST-	CONS-	REPEAT	(X,R OR X <sup>o</sup> )	(Y,S OR Y <sup>o</sup> )	(Z,T OR Z <sup>o</sup> )	(X,R OR X <sup>o</sup> )	(Y,S OR Y <sup>o</sup> )	(Z,T OR Z <sup>o</sup> )	*GENERATION*
NUMSER	SYST	NODE	RAINT	TRAIT	NODE	FORCE	FORCE	FORCE	MOMENT	MOMENT	MOMENT	NUMBER
49	6											INC
												COU-S
						0.	.10000E+01	0.	0.	0.	0.	0
												1
												1

CONCENTRATED FORCES (LOAD CASE 3)

HDR / URL BLOWDOWN MODEL

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LOAD CASE 3 --

NODE NUMBER	SPEC SYST	SKEW REST- NODE RAIN T	CONSTRAINT	REPEAT	(X,R OR X $\theta$ ) FORCE	(Y,S OR Y $\theta$ ) FORCE	(Z,T OR Z $\theta$ ) FORCE	(X,R OR X $\theta$ ) MOMENT	(Y,S OR Y $\theta$ ) MOMENT	(Z,T OR Z $\theta$ ) MOMENT	GENERATION $\theta$ NUMBER	INC COUNT
43	6				.10000E+01 0.	0.	0.	0.	0.	0.	0	1

C O N C E N T R A T E D F O R C E S (LOAD CASE 4)

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LOAD CASE 4 --

NODE NUMBER	SPEC SYST	SKEW NODE	REST- RAINT	CONS- RAINT	REPEAT NODE	(X,R OR X*) FORCE	(Y,S OR Y*) FORCE	(Z,T OR Z*) FORCE	(X,R OR X*) MOMENT	(Y,S OR Y*) MOMENT	(Z,T OR Z*) MOMENT	*GENERATION* NUMBER	INC	COUNT
37	5					.10000E+01	0.	0.	0.	0.	0.	0	1	1

C O N C E N T R A T E D F O R C E S (LOAD CASE 5)

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LOAD CASE 5 --

NODE NUMBER	SPEC SYST	SKEW REST- NODE RAIN T	CONS- RAINT	REPEAT NODE	(X,R OR X $\phi$ ) FORCE	(Y,S OR Y $\phi$ ) FORCE	(Z,T OR Z $\phi$ ) FORCE	(X,R OR X $\phi$ ) MOMENT	(Y,S OR Y $\phi$ ) MOMENT	(Z,T OR Z $\phi$ ) MOMENT	GENERATION $\phi$ NUMBER	INC COUNT
37	5				0.	.10000E+01 0.	0.	0.	0.	0.	0	1

C O N C E N T R A T E D F O R C E S (LOAD CASE 6)

HQZ / URL BLOWDOWN MODEL

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LOAD CASE 6 --

NODE NUMBER	SPEC SYST	SKEW REST- NODE RAIN	TRAIN	REPEAT	(X,R OR X#)	(Y,S OR Y#)	(Z,T OR Z#)	FORCE	(X,R OR X#)	(Y,S OR Y#)	(Z,T OR Z#)	MOMENT	MOMENT	MOMENT	GENERATION# NUMBER	INC	COUNT
18	4														0	1	1
								.10000E+01	0.	0.	0.	0.	0.	0.			



C O N C E N T R A T E D F O R C E S L O A D C A S E 8)

HDR / URL BLOWDOWN MODEL

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LOAD CASE 8 --

NODE NUMBER	SPEC SYST	SKW NODE	REST- RAINT	CONS- TRAIN	REPEAT NODE	(X,R OR X <sup>0</sup> ) FORCE	(Y,S OR Y <sup>0</sup> ) FORCE	(Z,T OR Z <sup>0</sup> ) FORCE	(X,R OR X <sup>0</sup> ) MOMENT	(Y,S OR Y <sup>0</sup> ) MOMENT	(Z,T OR Z <sup>0</sup> ) MOMENT	GENERATIONS NUMBER	INC COUNT
18	4					0.	0.	.1000E+01 0.	0.	0.	0.	0	1

C O N C E N T R A T E D F O R C E S (LOAD CASE 9)

HR / URL BLOWDOWN MODEL

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LOAD CASE 9 --

NODE NUMBER	SPEC SYST	SKW REST- NODE RATNT	TRSH?	REPEAT NODE	(X,R OR X $\theta$ ) FORCE	(Y,S OR Y $\theta$ ) FORCE	(Z,T OR Z $\theta$ ) FORCE	(X,R OR X $\theta$ ) MOMENT	(Y,S OR Y $\theta$ ) MOMENT	(Z,T OR Z $\theta$ ) MOMENT	GENERATION $\theta$ NUMBER	INC	COUNT
11	2				.1000E+01	0.	0.	0.	0.	0.	0	1	1



C O N C E N T R A T E D F O R C E S (LOAD CASE 10)

HDR / URL BLOWDOWN MODEL

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LOAD CASE 10 --

NODE NUMBER	SPEC SYST	SKEW NODE	REST- RAINT	CONS- TRAIT	REPEAT NODE	(X,R OR X*) FORCE	(Y,S OR Y*) FORCE	(Z,T OR Z*) FORCE	(X,R OR X*) MOMENT	(Y,S OR Y*) MOMENT	(Z,T OR Z*) MOMENT	*GENERATION* NUMBER	INC COUNT
11	2					0.	.10000E+01	0.	0.	0.	0.	0	1 1

J Y N A M I C J O B

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FLAG CONTROLLING THE METHOD USED TO  
CREATE THE SYSTEM MASS MATRIX = ( 1) IELMS (CC 16-20)  
EQ. 0, ELEMENT MASSES ARE NOT USED,  
ALL MASS DATA MUST BE INPUT IN  
THE \*MASSES\* DATA SECTION  
EQ. 1, ELEMENT MASSES ARE USED, ANY DATA  
GIVEN IN THE \*MASSES\* DATA SECTION  
WILL AUGMENT THE ELEMENT-BASED MATRIX

ACCELERATION OF GRAVITY = ( .98100E+01) FGEE (CC 21-30)

FLAG FOR PRINTING THE SYSTEM MASS MATRIX = ( 0) IMPRT (CC 31-35)  
EQ. 0, NO  
EQ. 1, YES

A-21

TIME HISTORY ANALYSIS CONTROL VARIABLES

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NUMBER OF SOLUTION TIME STEPS	= ( 5000)	IDT	(CC 16-20)
SOLUTION TIME STEP	= ( .20000E-03)	FDT	(CC 21-30)
TIME AT THE START OF THIS SOLUTION	= ( -.96600E-01)	FTIME0	(CC 31-40)
METHOD TO BE USED FOR DIRECT INTEGRATION EQ. N, NEWMARK'S METHOD EQ. 4, WILSON THETA METHOD	= ( N)	SN	(CC 41)
VALUE OF ALPHA PARAMETER (NEWMARK)	= ( .50000E+00)	FA	(CC 42-50)
VALUE OF BETA PARAMETER (NEWMARK)	= ( .25000E+00)	FB	(CC 51-60)
VALUE OF THETA PARAMETER (WILSON)	= ( 0.)	FT	(CC 61-70)
ANALYSIS TYPE (TRANSIENT OR STEADY-STATE)	= ( T)	ST	(CC 71)
PERIOD (STEADY-STATE ANALYSIS ONLY)	= ( 0.)	FTL	(CC 72-80)

\*\*\* ERROR \*\* PRECEDING LINE.  
SECTION (HISTORY), CARD(, 1), COLUMNS (31-40).  
ZERO (OR NEGATIVE) CONTROL PARAMETER.

PROCESSING TERMINATED FOR THIS SECTION.

DAMPING RATIOS

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REFERENCE DAMPING RATIO FOR ALL MODES = ( 0. ) FDAMP0 (CC 21-30)  
MASS PROPORTIONAL DAMPING CONSTANT = ( .12000E+01 ) FALPHA (CC 31-40)  
(DIRECT INTEGRATION OPTION ONLY)  
STIFFNESS PROPORTIONAL DAMPING CONSTANT = ( .63500E-04 ) FBETA (CC 41-50)  
(DIRECT INTEGRATION OPTION ONLY)

APPENDIX B

PRESSURE WAVE FLUID FORCES ON URL PIPE

## APPENDIX B

### PRESSURE WAVE FLUID FORCES ON URL PIPE

The approach used to determine the fluid forces on the URL pipe was to assume the value of the momentum terms in the linear momentum equation, from fluid mechanics (control volume formulation), to be small compared to the surface force terms. For this reason, the momentum terms, together with the body force term, were neglected for this analysis. The resulting equation to be solved for the fluid force on the pipe  $\bar{F}_R$  is  $\bar{F}_S = 0$ . The solution is obtained simply as follows (see Figure B.1):

$$\bar{F}_S = -\bar{F}_R + p_1 A_1 \hat{i}_1 - p_2 A_2 \hat{i}_2 = 0 \quad (\text{sum of the surface forces on the C.S.})$$

$$\bar{F}_R = p_1 A_1 \hat{i}_1 - p_2 A_2 \hat{i}_2, \quad (\text{B-1})$$

where  $\bar{F}_R$  is the resultant force of a control volume on its corresponding section of pipe. This equation is applied, as follows, to the control volumes defined for this solution approach.

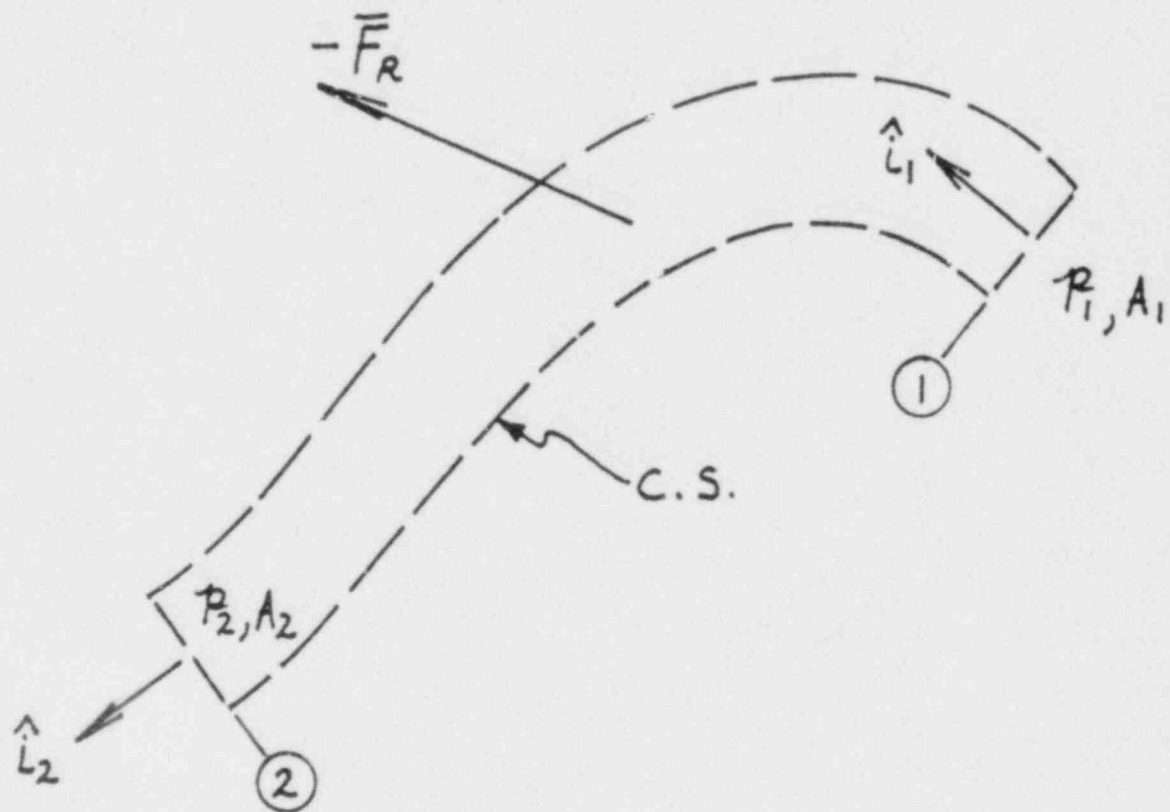


FIGURE B.1: Pipe Force on Control Volume.

<b>ANCO</b> ANCO Engineers, Incorporated 1701 Colorado Avenue, Santa Monica, CA 90404 (213) 829-9721, 829-2624	DESCRIPTION <u>GSP4a</u>
	CALCULATIONS FOR <u>1182-B (NRC)</u>
MADE BY <u>WBN</u> DATE <u>7/21/81</u>	
CHECKED BY <u>GEH</u> DATE <u>9/11/81</u>	

## Force of Fluid (Control Volumes) On Pipe

Apply equation B-1 as follows (use Figures B.2-B.7):

$$\begin{aligned}
 LP_1 : \quad \bar{F}_1 &= P_1 A_1 \sin 29^\circ (-\hat{i}) + P_1 A_1 \cos 29^\circ \hat{j} \\
 &\quad + P_2 A_2 \hat{i} \\
 &= (P_2 A_2 - P_1 A_1 \sin 29^\circ) \hat{i} + P_1 A_1 \cos 29^\circ \hat{j}
 \end{aligned}$$

$$\begin{aligned}
 LP_2 : \quad \bar{F}_2 &= -P_2 A_2 \hat{i} + P_3 A_3 \hat{i} \\
 &= (P_3 A_3 - P_2 A_2) \hat{i}
 \end{aligned}$$

$$\begin{aligned}
 LP_3 : \quad \bar{F}_3 &= P_4 A_4 \hat{j} + P_5 A_5 \hat{i} \\
 &= P_5 A_5 \hat{i} + P_4 A_4 \hat{j}
 \end{aligned}$$

$$\begin{aligned}
 LP_4 : \quad \bar{F}_4 &= P_6 A_6 \hat{k} + P_7 A_7 \cos 60^\circ \hat{i} + P_7 A_7 \sin 60^\circ \hat{j} \\
 &= P_7 A_7 \cos 60^\circ \hat{i} + P_7 A_7 \sin 60^\circ \hat{j} + P_6 A_6 \hat{k}
 \end{aligned}$$





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DESCRIPTION	GSP4a
CALCULATIONS FOR	1182-B (NRC)

MADE BY	WBW	DATE	7/21/81
CHECKED BY	[Signature]	DATE	9/11/81

$$LP_5: \quad \bar{F}_5 = -P_7 A_7 \cos 15^\circ \hat{i} - P_7 A_7 \sin 15^\circ \hat{j}$$

These equations were implemented in solving GSP4a. They were coded (programmed) in the computer code PWFORC. A listing of this code is given at the end of this appendix.

**ANCO**

ANCO Engineers, Incorporated  
 1701 Colorado Avenue, Santa Monica, CA 90404  
 (213) 829-9721, 829-2624

DESCRIPTION GSP4A  
 CALCULATIONS FOR 11B2-B (NRC)

MADE BY WBW DATE 7/21/81  
 CHECKED BY \_\_\_\_\_ DATE \_\_\_\_\_

Figure B.2

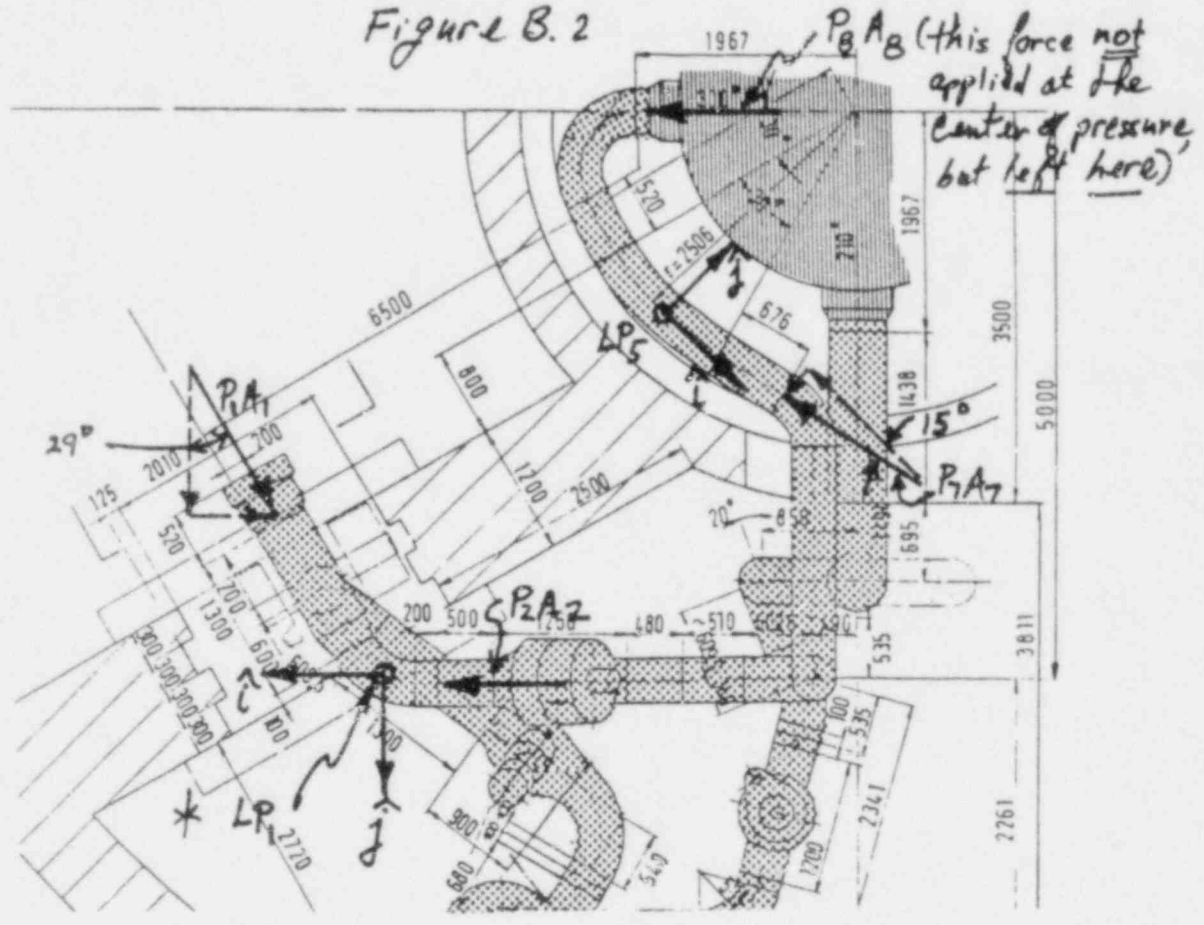
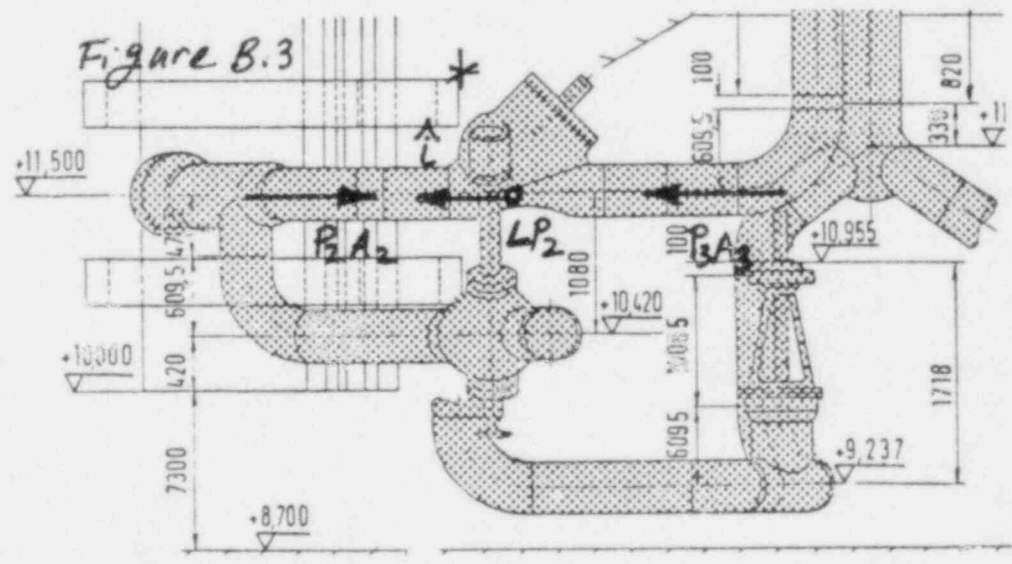


Figure B.3

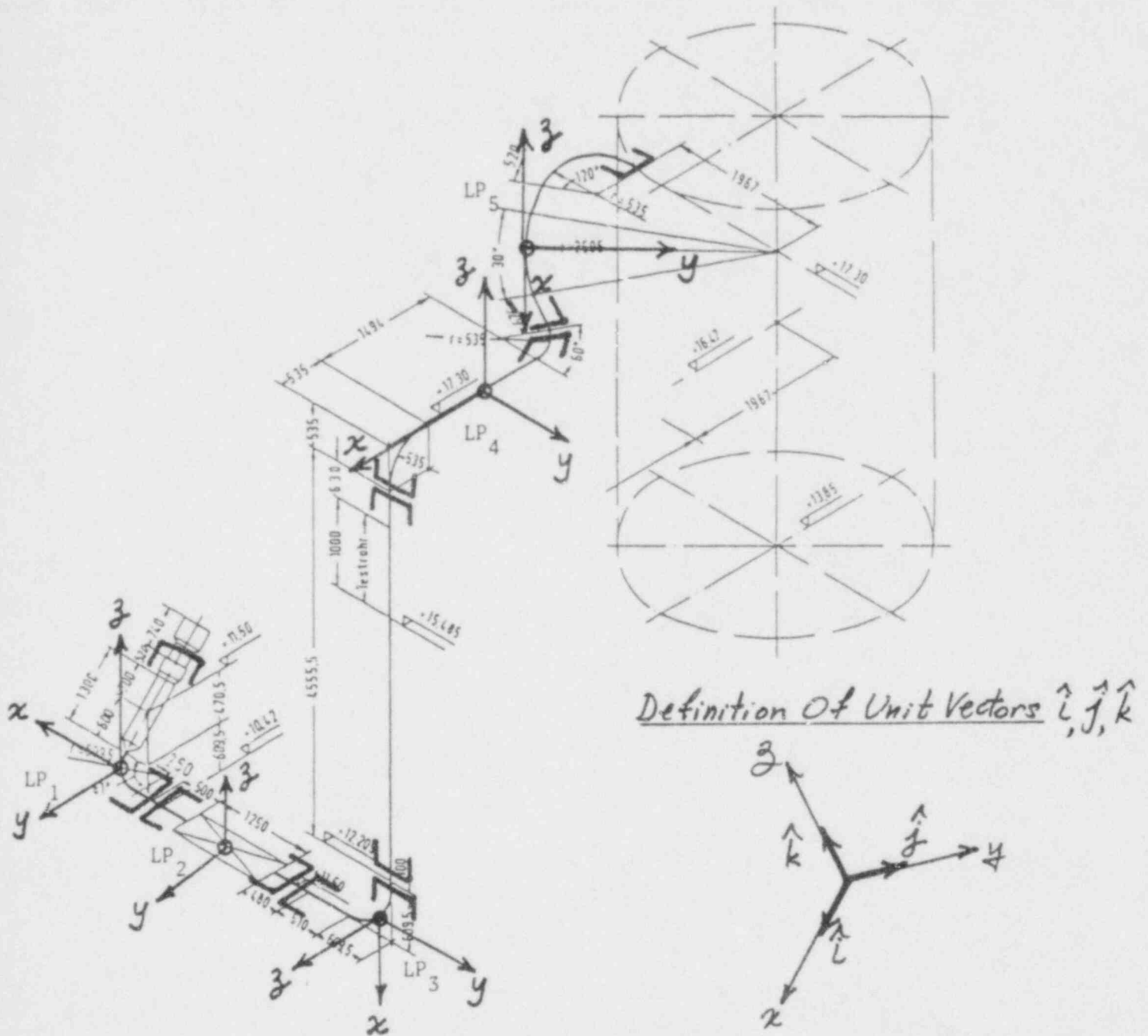


\* Note: See Figure B.7 concerning the definition of the unit vectors  $i, j, k$ . B-6



# Figure B.7

LOCAL COORDINATE DIRECTIONS FOR APPLIED FLUID FORCES



$LP_j$  = load point j (corresponds to control volume c.v. j)

[ ] = bracket pair designates a control volume

```

C *****
C PROGRAM PAFORC INPUT, OUTPUT, TAPES=INPUT, TAPE6=OUTPUT, TAPE7, TAPE8)
C *****
C *** PAFORC ***
C
C THIS PROGRAM CALCULATES THE PRESSURE WAVE FORCES ON A PIPING
C SYSTEM. THE PRESSURES ARE OBTAINED FROM EXPERIMENTAL PRESSURE
C DATA. THIS PROGRAM APPLIES SPECIFICALLY TO THE JAL BLOKOWAN
C LEG(MOR, WEST GERMANY). THE LEG IS FIXED AT THE REACTOR VESSE-
C WALL AND AT THE TEE. IT IS A SINGLE RUN OF PIPE WITH NO
C BRANCHES AND NO SUPPORTS(HANGERS, ETC.). THE FORCES ARE
C DETERMINED FOR FIVE(S) LOAD POINTS. TEN LOADS ARE DETERMINED.
C THE PRESSURES ARE REPRESENTED BY THE ARRAY P.
C *****
C
C DIMENSION P(7,5000),A(7),F(10,5000),ITITLE(40)
C DIMENSION TIME(5000),IMHERE(10),DT(5000)
C DIMENSION POUT(7)
C
C REMIND 7
C REMIND 8
C
C ** READ INPUT DATA **
C * READ DATA FROM CARDS
C READ(5,10) ITITLE
C READ(5,11) N1PTS
C READ(5,12) A
C READ(5,400) IMHERE
C FORMAT(10I5)
C
C *CALL SUBROUTINE DOSYSX TO READ GERMAN DATA IN DOSYS FORMAT
C CALL DOSYSX(TIME,P,5000,7,IMHERE,N1PTS,8)
C ** END OF READING DATA **
C
C * BIAS THE PRESSURES
C DO 999 J=1,7
C P(J) = P(J,1)
C DO 1000 I=1,N1PTS
C DO 1000 J=1,7
C P(J,IT) = P(J,IT) - P(J)
C
C ** CALC. AVERAGE TIME INCREMENT(ADT)**
C N1 = N1PTS - 1
C DO 410 I=1,N1
C DT(I) = TIME(I+1) - TIME(I)
C ADT = 0.0
C DO 420 I=1,N1
C ADT = ADT + DT(I)
C ADT = ADT/N1
C ** COMPARE DT(I) WITH ADT **
C DIFMAX = -1.0
C DO 430 I=1,N1
C DIF = ABS(DT(I) - ADT)
C IF(DIF.GT.DIFMAX) DIFMAX = DIF
C CONTINUE
C
C ** CALCULATE FORCES OF FLUID ON U(RL)D) PIPE(PRESSURE WAVE ONLY)
C *
C FORCE INFORMATION
C LOAD
C POINT ANCO FORCE LP COORD-
C ILP) NODE AT LP DIRECTION
C 1 49 F(1,1) X
C 1 49 F(2,1) Y
C 2 49 F(3,1) X

```

```

C      3      37  F(4,I)   X
C      3      37  F(5,I)   Y
C      4      18  F(6,I)   X
C      4      18  F(7,I)   Y
C      4      18  F(8,I)   Z
C      5      11  F(9,I)   X
C      5      11  F(10,I)  Y
C      * FORCES F(1,I) THRU F(10,I) ARE COMPONENTS IN LOCAL COORD. SYS.5

```

\* PRESSURE INFORMATION

```

C      ANCO
C      NODE  PRESSURE  AREA(M**2)
C      51    P(1,I)   A(1) = 0.1083
C      46    P(2,I)   A(2) = 0.1083
C      43    P(3,I)   A(3) = 0.1083
C      40    P(4,I)   A(4) = 0.1083
C      36    P(5,I)   A(5) = 0.1018
C      22    P(6,I)   A(6) = 0.1018
C      14    P(7,I)   A(7) = 0.1018

```

(NOTE-- IN THE ABOVE - I - IS INTEGER TIME.)

(NOTE--ANCO NODES(NO.5) ARE THOSE FROM THE EASEZ MODEL FEB81)

\* THE AREA A IN THE EQ.S FOR F IS ACTUALLY A\* = 10\*\*5\*A(M\*\*2)

AND HAS UNITS N/BAR.

\* THE UNITS OF P ARE BAR.

\* THE UNITS OF F ARE N.

\* INITIALIZE ARRAY F

NTPTSS = (INT((NTPYS-0.5)/8)+1)\*8

DO 300 K=1,10

DO 300 I=1,NTPTSS

300 F(K,I) = 1.0

\* CALC. ARRAY F

DO 100 IT=1,NTPTS

F(1,IT) = P(2,IT)\*A(2) - 0.485\*P(1,IT)\*A(1)

F(2,IT) = 0.875\*P(1,IT)\*A(1)

F(3,IT) = P(3,IT)\*A(3) - P(2,IT)\*A(2)

F(4,IT) = P(5,IT)\*A(5)

F(5,IT) = P(4,IT)\*A(4)

F(6,IT) = 0.500\*P(7,IT)\*A(7)

F(7,IT) = 0.866\*P(7,IT)\*A(7)

F(8,IT) = P(6,IT)\*A(6)

F(9,IT) = -0.966\*P(7,IT)\*A(7)

F(10,IT) = -0.259\*P(7,IT)\*A(7)

100 CONTINUE

\*\* END CALCULATING FORCES \*\*

\*\* WRITE OUT DATA \*\*

\* WRITE THE FORCES OUT TO TAPE7(EASEZ FORMAT)

LTAPE = 7

DO 150 K=1,10

WRITE(LTAPE,207) (F(K,I),I=1,NTPTSS)

ENDFILE LTAPE

150 CONTINUE

\* WRITE OUT TO OUTPUT

WRITE(6,200)

WRITE(6,201) TITLE

WRITE(6,202) NTPTS

WRITE(6,440) TIME(1),AGF,DIFMAX

```

440 FORMAT 1X,16#ADDITIONAL INFO,/,1X,
*      BHTIME = E12.4,/,1X,
*      10#AVG. DT = E12.4,/,1X,
*      13#MAX.VAR.DT = E12.4,/,/
      WRITE(6,203) A
      WRITE(6,217) IMHERE
217 FORMAT(1X,15#IMHERE ARRAY = ,1019)
      WRITE(6,998) P0
998 FORMAT(//,1X,23#BIASING PRESSURES °C = ,7E12.3)
      DO 210 K=1,7
      WRITE(6,204) K
      WRITE(6,205) (P(K,I),I=21,NTPTS,20)
210 CONTINUE
      DO 211 K=1,10
      WRITE(6,206) K
      WRITE(6,205) (P(K,I),I=21,NTPTS,20)
211 CONTINUE
C      * * END OF WRITING OUT * *
C      * * FORMATS * *
C
10 FORMAT(40A2)
11 FORMAT(10I5)
12 FORMAT(8F10.4)
200 FORMAT(11H,47#PROGRAM P#FORC -- CALCULATES THE PRESSURE WAVE *
*      17#FORCE ON URL PIPE,/)
201 FORMAT(11H,40A2,/)
202 FORMAT(11H,23#NTPTSING. TIME PT.S) = ,19,/)
203 FORMAT(11H,27#FLOW AREA$A1,A2,....,A7) = ,7E12.3,/)
204 FORMAT(11H,24#PRESSURE TIME HISTORY P(,11,1H),/)
205 FORMAT(1X,10E12.3)
206 FORMAT(11H,21#FORCE TIME HISTORY F(,12,1H))
207 FORMAT(8E10.4)
C      * * END FORMATS * *
      STOP
      END

```







```

READLICHAN,9020) TSTAT
WRITE(6 ,9120) TSTAT
200 CONTINUE
C
C READ INPUT DATA AND SAVE REQUESTED PRESSURES
C DONT READ PRESSURES IF IMHERE(1)=0.
C
IF(IMHERE(1)=EQ=0) GOTO 500
DD ADD I=1,NSAMP
READLICHAN,9021) TIME(I),IDINPUT(J),J=1,NTRAN)
DO 300 J=1,NPSR
PRESSR(I,J)=DINPUT(IMHERE(J))
300 CONTINUE
400 CONTINUE
500 CONTINUE
NPTS = NSAMP
RETURN
1000 CONTINUE
WRITE(6,9000)
STOP
C
C FORMATS
C
9000 FORMAT(1X,31H UNABLE TO HANDLE NO. OF POINTS)
9001 FORMAT(16,74X)
9002 FORMAT(16,75X)
9003 FORMAT(10,A2,68X)
9004 FORMAT(1,A3,211,12,16,66X)
9006 FORMAT(2A10,A4,56X)
9007 FORMAT(80X)
9017 FORMAT(15,75X)
9019 FORMAT(16,A2,A6,A1,A10,A4,A3,315,A1,324)
9020 FORMAT(2A10,A8,52X)
9101 FORMAT(16L2,5,8X)
9102 FORMAT(1X,16)
9103 FORMAT(1X,46)
9104 FORMAT(1X,A1,A3,211,12,16)
9106 FORMAT(1X,2A10,A4)
9107 FORMAT(1X,15)
9119 FORMAT(1X,A6,A2,A6,A1,A10,A4,A3,315,A1)
9120 FORMAT(1X,2A10,A8)
END
005V1630
005V1640
005V1650
005V1660
005V1670
005V1680
005V1690
005V1700
005V1710
005V1720
005V1730
005V1740
005V1750
005V1760
005V1770
005V1780
005V1790
005V1800
005V1810
005V1820
005V1830
005V1840
005V1850
005V1860
005V1870
005V1880
005V1890
005V1900
005V1910
005V1920
005V1930
005V1940
005V1950
005V1960
005V1970
005V1980
005V1990
005V2000
005V2010

```

APPENDIX C

PROCESSING OF OUTPUT FROM EASE2 AND  
COMPUTATION OF ADDITIONAL STRESSES

## APPENDIX C

### PROCESSING OF OUTPUT FROM EASE2 AND COMPUTATION OF ADDITIONAL STRESSES

The computer code PWSTRS was written to apply coordinate transformations to the displacement and acceleration output from EASE2, compute additional stress components, change to desired units (i.e., displacements were to have units of mm), and format solution results in DOSYS format. A listing of this code is given in this appendix.





```

C          QUANTITY
C          KEY .EQ.1,2      * NODE NUMBER
C          KEY .EQ.3,4,5,6,7 * ELEMENT NUMBER
C          S3(J).EQ.9H      ACS. ACS NUMBER
C
5  READ(EZHIST)KEY,NOS,ITPH,MAXCOL,
C      (S1(1,J),S1(2,J),J=1,MAXCOL),
C      (S2(1,J),S2(2,J),J=1,MAXCOL),
C      (S3( J),L3RN(J),J=1,MAXCOL)
      IF(EOF(EZHIST))GOO,80
80  CONTINUE
      WRITE(6,302) KEY,NOS,ITPH
302  FORMAT(1X,24HINFO. FROM HEADER RECORD,/,
C      *      9X,20HCATEGORY NO.(KEY) = ,15,/ ,
C      *      9X,32HOUTPUT SET NO. IN KEY-TH CAT. = ,15,/ ,
C      *      9X,18HOUTPUT INTERVAL = ,15,/)
      WRITE(6,303) S3(1),L3RN(1)
303  FORMAT(9X,A9,I5,/)
      L = L3RN(1)
C      ** BRANCHING,DEPENDENT UPON TYPE OF DATA
      IF(KEY.EQ.1) GOTO 51
      IF(KEY.EQ.2) GOTO 53
      IF(KEY.EQ.7) GOTO 52
C      ** READ AND SET UP NODE DATA FOR OUTPUT(DISPL.S AND ACCEL.S)
51  CONTINUE
      IF(L.EQ.1)IND=1
      IF(L.EQ.2)IND=2
      IF(L.EQ.42)IND=1
      IF(L.EQ.43)IND=3
C      (IND REFERS TO THE INTERNAL NODE NUMBERS FOR OUTPUT OF NODE
C      DATA(KEY=1).)
      WRITE(6,304) L,IND
304  FORMAT(1X,32HINFO. FROM COMP. SEQ.(NODE DATA),/,
C      *      9X,20HEXTERNAL NODE NO. = ,15,/ ,
C      *      9X,20HINTERNAL NODE NO. = ,15,/)
      GOTO 10
C      ** READ AND SET UP PIPE DATA FOR OUTPUT(STRESSES)
52  CONTINUE
      IF(L.EQ.1)IND=1
      IF(L.EQ.7)IND=2
      IF(L.EQ.15)IND=3
      IF(L.EQ.17)IND=4
      IF(L.EQ.21)IND=5
C      (IND REFERS TO THE INTERNAL NODE NUMBERS FOR OUTPUT OF PIPE
C      DATA(KEY=7).)
      WRITE(6,305) L,IND
305  FORMAT(1X,32HINFO. FROM COMP. SEQ.(PIPE DATA),/,
C      *      9X,20HEXTERNAL PIPE NO. = ,15,/ ,
C      *      9X,20HINTERNAL NODE NO. = ,15,/)
      GOTO 10
C      ** READ AND SET UP FORCE DATA FOR OUTPUT (FORCES)
53  CONTINUE
      IF(L.EQ.11)IND=1
      IF(L.EQ.18)IND=2
      IF(L.EQ.37)IND=3
      IF(L.EQ.4)IND=4
      IF(L.EQ.49)IND=5
C      (IND REFERS TO THE INTERNAL NODE NUMBERS FOR OUTPUT OF FORCE
C      DATA(KEY=2).)
      WRITE(6,306) L,IND
306  FORMAT(1X,33HINFO. FROM COMP. SEQ.(FORCE DATA),/
C      *      9X,20HEXTERNAL NODE NO. = ,15,/ ,
C      *      9X,20HINTERNAL NODE NO. = ,15,/)

```

DEFINITION OF INTERNAL NODE NUMBERS(IND) (2/13/81)

IND REFERS TO A NUMBERING SYSTEM WITHIN THIS CODE THAT CORRESPONDS TO AN EXTERNAL(PHYSICAL) NUMBERING SYSTEM OF NODE AND ELEMENT NUMBERS.

EXTERNAL* NODE NUMBER	IND	NODE DATA CORRESPONDING RESPONSE QUANT.
14	1	DISPL(3.4M VOM RDB-STUTZEN)
22	2	DISPL(7.7M VOM RDB-STUTZEN)
42	1	ACCELA(M VENTIL)
43	3	DISPL(A M VENTIL)

EXTERNAL* ELEMENT NUMBER	IND	ELEMENT DATA CORRESPONDING RESPONSE QUANT.
1	1	STRESS(MESSEBENE A)
7	2	STRESS(MESSEBENE C1)
15	3	STRESS(MESSEBENE M1)
17	4	STRESS(MESSEBENE E)
21	5	STRESS(MESSEBENE F)

EXTERNAL* NODE NUMBER	IND	FORCE DATA CORRESPONDING RESPONSE QUANT.
11	1	FORCE
18	2	FORCE
37	3	FORCE
43	4	FORCE
49	5	FORCE

\* NOTE- THESE ARE THE EASE2 NODE AND ELEMENT NUMBERS(FEB 81)

EZHIST D A T A

VARIABLES

TIME = TIME AT WHICH HISTORY RESULTS ARE SAVE: ON THE EZHIST FILE

XI(IJ) = VALUE OF THE J-TH REQUEST QUANTITY AT SOLUTION TIME = TIME--.

\* SET I = 1 (INTEGER TIME = 1). THIS IS FOR LOOP FROM STATEMENT 10 TO 20.

10 I = 1

```

DO 20 ILOOP=1,10
IF(MOD(I,I*PHI).NE.0) GOTO 20
READ(EZHIST) TIME,(XI(IJ),J=1,MAXCOL)
IF(I*SKIP.EQ.1) GOTO 60
** GENERATE ARRAY OF SOLUTION TIMES
TIME(I) = TIME
WRITE(ITTPE) TIME(I)

```

60 CONTINUE

\*\* BRANCHING DEPENDENT UPON TYPE OF DATA

IF(KEY.EQ.1) GOTO 55

IF(KEY.EQ.2) GOTO 57

IF(KEY.EQ.7) GOTO 56

\*\* NODE DATA FOR OUTPUT

55 CONTINUE

IF(L.EQ.42) GOTO 70

\* DISPLACEMENT DATA IN THE FOLLOWING D = TD\*XI , WHERE D ARE THE LOCAL

```

C      DISPLACEMENTS, TD IS THE TRANSFORMATION MATRIX, AND XI ARE THE GLOBAL
C      DISPLACEMENTS.
      DX(IND,1) = TD(1,1)*XI(1) + TD(1,2)*IND)*XI(2) +
      * TD(1,3)*IND)*XI(3))/1000.0
      DY(IND,1) = TD(2,1)*XI(1) + TD(2,2)*IND)*XI(2) +
      * TD(2,3)*IND)*XI(3))/1000.0
      DZ(IND,1) = TD(3,1)*XI(1) + TD(3,2)*IND)*XI(2) +
      * TD(3,3)*IND)*XI(3))/1000.0
C      [DISPLACEMENT AT IND IS (DX,DY,DZ), UNITS ARE MM.]
      DTAPE = 10 + IND
      WRITE(OTAPE) DX(IND,1),DY(IND,1),DZ(IND,1)
      GO TO 20
C 70 CONTINUE
      * ACCELERATION DATA A = TA*XI
      AX(IND,1) = (TA(1,1)*XI(1) + TA(1,2)*XI(2) +
      * TA(1,3)*XI(3))/9.81
      AY(IND,1) = (TA(2,1)*XI(1) + TA(2,2)*XI(2) +
      * TA(2,3)*XI(3))/9.81
      AZ(IND,1) = (TA(3,1)*XI(1) + TA(3,2)*XI(2) +
      * TA(3,3)*XI(3))/9.81
C      [ACCELERATION AT IND IS (AX,AY,AZ), UNITS ARE G-S.]
      ATAPE = 13 + IND
      WRITE(ATAPE) AX(IND,1),AY(IND,1),AZ(IND,1)
      GO TO 20
C 56 CONTINUE
      * PIPE DATA FOR OUTPUT
      IF(MXCOL.EQ.2) GO TO 75
      STPB(IND,1) = XI(1)/1.0E6
      STEN(IND,1) = XI(5)/1.0E6
      SBEN(IND,1) = XI(6)/1.0E6
      STDR(IND,1) = XI(7)/1.0E6
      * PERFORM CALC. S WITH EASEZ DATA AND PRESSURE(P) DATA
      IPL = ILOOP + 1
      GO TO (681,682,683,684,685),IND
      681 SPAL(1) = 0.0
      SPT(1,1) = 0.0
      GO TO 686
      682 SPA(2,1) = 0.0
      SPT(2,1) = 0.0
      GO TO 686
      683 P(5,1) = P(5,1) - P(5,1)
      SPA(3,1) = 0.10*P(5,1)/R2ML(5)
      SPT(3,1) = 2*SPA(3,1)
      GO TO 686
      684 P(4,1) = P(4,1) - P(4,1)
      SPA(4,1) = 0.10*P(4,1)/R2ML(4)
      SPT(4,1) = 2*SPA(4,1)
      GO TO 686
      685 P(2,1) = P(2,1) - P(2,1)
      SPA(5,1) = 0.10*P(2,1)/R2ML(2)
      SPT(5,1) = 2*SPA(5,1)
      CONTINUE
C      [SPA AND SPT ARE THE AXIAL AND TANGENTIAL STRESS DUE TO
C      INTERNAL PRESSURE(P), RESPECTIVELY, THEY ARE AT INTERVAL
C      NODE POINTS(IND) 3,4, AND 5. UNITS OF STRESS ARE N/MM**2.]
      J = IND
      SA(1,1) = STPB(1,1) + SPA(1,1)
      SV(1,1) = SRT(SA(1,1)**2 + SPT(1,1)**2 - SA(1,1)*SPT(1,1)
      * 3*STDR(1,1)**2)
C      (STPB+STEN+SBEN+STDR, ARE THE TENSILE + BENDING, TENSILE +
C      BENDING, AND TORSIONAL STRESSES, RESPECTIVELY, UNITS ARE
C      N/MM**2.)
C      [SV IS THE COMPARATIVE STRESS, UNITS ARE N/MM**2.]

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```

TAPE = 14 * IND
WRITE(TAPE) STEN(IND,1),SBEN(IND,1),STOR(IND,1),SP(IND,1),
* SV(IND,1)
GOTO 20
75 CONTINUE
MY(IND,1) = XI(1)
MZ(IND,1) = XI(2)
J = IND
ZEROM0 = 0.001
MY = ABS(MY(J,1))
MZ = ABS(MZ(J,1))
MZ5 = MZ(J,1)
IF(MY.LT.ZEROM0) GOTO 929
ALPHA(J,1) = ATAN2(MZ(J,1),MY(J,1))*57.3
GOTO 927
929 CONTINUE
IF(MZ.LT.ZEROM0) GOTO 928
IF(MZ5.GT.0.0) ALPHA(J,1) = 90.0
IF(MZ5.LT.0.0) ALPHA(J,1) = -90.0
GOTO 927
928 CONTINUE
ALPHA(J,1) = 0.0
927 CONTINUE
C IMY AND MZ ARE THE Y AND Z BENDING MOMENTS, UNITS ARE *N.*
C (ALPHA IS THE ANGLE OF THE BENDING AXIS, UNITS ARE DEGREES.)
ALTAPE = 19 * IND
WRITE(ALTAPE) ALPHA(IND,1)
GOTO 20
* * FORCE DATA FOR OUTPUT ( F = TF*XI )
57 CONTINUE
FX(IND,1) = TF(1,1,IND)*XI(1) + TF(1,2,IND)*XI(2) *
FY(IND,1) = TF(2,1,IND)*XI(1) + TF(2,2,IND)*XI(2) *
FZ(IND,1) = TF(3,1,IND)*XI(1) + TF(3,2,IND)*XI(2) *
(FORCE AT IND IS (FX,FY,FZ), UNITS ARE N.)
FTAPE = 24 * IND
WRITE(FTAPE) FX(IND,1),FY(IND,1),FZ(IND,1)
C * IN THIS LOOP10 TO 20) I = 1 .
C
C 20 CONTINUE
ISKIP = 1
* * TRANSFER TO -5- TO READ NEXT HEADER RECORD
GOTO 5
1000 CONTINUE
C (ALL ITEMS HAVE BEEN CALCULATED.)
C
C * REMIND TAPES 10 THRU 29
DO 679 J=10,29
679 REMIND J
C * * WRITE OUT TO OUTPUT * *
C * FOR OUTPUT TO PRINTER VIA TAPE I = 1 .
I = 1
WRITE(6,307)
307 FORMAT(1H1,43HOUTPUT CALC. RESULTS(OUTPUT EVERY * * STEPS),////)
331 FORMAT(1X,10HTIME ARRAY,///)
K = 0

```

```

DO 600 J=1,107
READ(TAPE) TTIME(I)
K = K + 1
IF(K.EQ.M8) GOTO 610
GOTO 600
K = 0
610 WRITE(6,332) TTIME(I)
600 CONTINUE
332 FORMAT(3X,8E15.4)
333 WRITE(6,333)
334 FORMAT(1H1)
335 WRITE(6,308)
308 FORMAT(1X,13HDISPLACEMENTS,/)
DO 310 IND=1,3
WRITE(6,309) IND
309 FORMAT(1H,6HIND = ,15,/,8X,2HDX,14X,2HDX,14X,2HDX,14X,2HDX,/)
K = 0
DTAPE = 10 + IND
DO 611 J=1,107
READ(TAPE) DX(IND,1),DY(IND,1),DZ(IND,1)
K = K + 1
IF(K.EQ.M2) GOTO 611
GOTO 611
K = 0
611 WRITE(6,311) DX(IND,1),DY(IND,1),DZ(IND,1)
611 CONTINUE
310 CONTINUE
311 FORMAT(3E12.4,AX)
312 FORMAT(1H,13HACCELERATIONS,/)
WRITE(6,313)
313 FORMAT(1H,7HIND = 1,/,8X,2HAX,14X,2HAY,14X,2HAZ,/)
IND = 1
ATAPE = 13 + IND
DO 642 J=1,107
READ(TAPE) AX(IND,1),AY(IND,1),AZ(IND,1)
K = K + 1
IF(K.EQ.M2) GOTO 612
GOTO 642
K = 0
612 WRITE(6,311) AX(1,1),AY(1,1),AZ(1,1)
642 CONTINUE
314 FORMAT(1H,8HSTRESSES,/)
DO 315 IND=1,5
WRITE(6,316) IND
316 FORMAT(1H,6HIND = ,15,/,8X,4HSTEN,14X,4HSEBEN,13X,5HALP=4,14X,
4HSTOR=14X,3HSPT,14X,4H SV,/)
K = 0
DTAPE = 14 + IND
ATAPE = 19 + IND
DO 643 J=1,107
READ(TAPE) STEN(IND,1),SEBEN(IND,1),STOR(IND,1),SPT(IND,1),
SV(IND,1)
* READ(TAPE) ALP:(IND,1)
K = K + 1
IF(K.EQ.M2) GOTO 613
GOTO 643
K = 0
613 WRITE(6,317) STEN(IND,1),SEBEN(IND,1),ALPHA(IND,1),STOR(IND,1),
SPT(IND,1),SV(IND,1)
* 643 CONTINUE

```

```

315 CONTINUE
317 FORMAT( 6E12.4,6X)
WRITE(6,320)
320 FORMAT(1H1, 6HFORCES,/)
DO 321 IND=1,5
WRITE(6,321) IND
322 FORMAT(1H, 6HIND = ,15,/,8X,2HF,14X,2HFY,14X,2HFZ,/)
K = 0
FTAPE = 24 + IND
DO 644 J=1,101
READ(TAPE) FX(IND,1),FY(IND,1),FZ(IND,1)
K = K + 1
IF(EG.M2) GOTO 614
GOTO 644
614 K = 0
WRITE(6,311) FX(IND,1),FY(IND,1),FZ(IND,1)
644 CONTINUE
321 CONTINUE
C
C
C
C
** OUTPUT DATA TO TAPE USING DOSYS FORMAT
* GENERATE DATA FOR FILE1 (DOSYS FORMAT)
DO 698 J=10,29
REIND J
TIMSTP = 0.0002
NTRAN = 42
ICHAN = 30
DO 433 J=1,42
IMH(J) = J
DATE = 6810327
CALL DOSYS(TIME,TIMSTP,NTRAN,101,XOUT,42,ICHAN,IMH,IDATE,0)
* SET I = 1 FOR GEN. GERMAN OUTPUT.
I = 1
DO 500 ILOOP = 1,101
*READ(TAPE) TTIME(1)
DO 400 IND=1,3
DTAPE = 10 + IND
400 READ(TAPE) DX(IND,1),DY(IND,1),DZ(IND,1)
IND = 1
ATAPE = 13 + IND
READ(TAPE) AX(IND,1),AY(IND,1),AZ(IND,1)
DO 401 IND=1,5
STAPE = 14 + IND
401 READ(STAPE) STEN(IND,1),SBEN(IND,1),STOR(IND,1),SPT(IND,1),
* SV(IND,1)
DO 402 IND=1,5
ALTAPE = 19 + IND
READ(ALTAPE) ALPHA(IND,1)
TTIME(1) = TTIME(1) - 0.0966
XOUT(1,1) = DY(1,1)
XOUT(1,2) = DX(1,1)
XOUT(1,3) = DZ(1,1)
XOUT(1,4) = DY(2,1)
XOUT(1,5) = DZ(2,1)
XOUT(1,6) = DX(2,1)
XOUT(1,7) = DZ(3,1)
XOUT(1,8) = DX(3,1)
XOUT(1,9) = DY(3,1)
XOUT(1,10) = AX(1,1)
XOUT(1,11) = AY(1,1)
XOUT(1,12) = AZ(1,1)
XOUT(1,13) = STEN(1,1)

```

```

XOUT(1,14) = SBEN(1,1)
XOUT(1,15) = ALPHA(1,1)
XOUT(1,16) = STOR(1,1)
XOUT(1,17) = 0.0
XOUT(1,18) = 0.0
XOUT(1,19) = STEN(2,1)
XOUT(1,20) = SBEN(2,1)
XOUT(1,21) = ALPHA(2,1)
XOUT(1,22) = STOR(2,1)
XOUT(1,23) = 0.0
XOUT(1,24) = 0.0
XOUT(1,25) = STEN(3,1)
XOUT(1,26) = SBEN(3,1)
XOUT(1,27) = ALPHA(3,1)
XOUT(1,28) = STOR(3,1)
XOUT(1,29) = SPT(3,1)
XOUT(1,30) = SV(3,1)
XOUT(1,31) = STEN(4,1)
XOUT(1,32) = SBEN(4,1)
XOUT(1,33) = ALPHA(4,1)
XOUT(1,34) = STOR(4,1)
XOUT(1,35) = SPT(4,1)
XOUT(1,36) = SV(4,1)
XOUT(1,37) = STEN(5,1)
XOUT(1,38) = SBEN(5,1)
XOUT(1,39) = ALPHA(5,1)
XOUT(1,40) = STOR(5,1)
XOUT(1,41) = SPT(5,1)
XOUT(1,42) = SV(5,1)
CALL DOSYSWITTIME,TIMSTP,NTRAN,1,XOUT,42,ICHAN,IWH, IDATE,1)

```

500 CONTINUE

```

* GENERATE DATA FOR FILE2 (DOSYS FORMAT)

```

```

REWIND 10
NTRAN = 15
ICHAN = 31
CALL DOSYSWITTIME,TIMSTP,NTRAN,1,XOUT,42,ICHAN,IWH, IDATE,0)

```

```

* SET I = 1 FOR OUTPUT OF FORCES.

```

```

I = 1
DO 595 ILOOP=1,10
READ(1TAPE) TTIME(1)
DB 530 IND=1,5
FTAPE = 24 + IND
530 READ(FTAPE) FX(IND,1),FY(IND,1),FZ(IND,1)

```

```

TTIME(1) = TTIME(1) - 0.0966

```

```

XOUT(I, 1) = FX(1,1)
XOUT(I, 2) = FY(1,1)
XOUT(I, 3) = FZ(1,1)
XOUT(I, 4) = FX(2,1)
XOUT(I, 5) = FY(2,1)
XOUT(I, 6) = FZ(2,1)
XOUT(I, 7) = FX(3,1)
XOUT(I, 8) = FY(3,1)
XOUT(I, 9) = FZ(3,1)
XOUT(I,10) = FX(4,1)
XOUT(I,11) = FY(4,1)
XOUT(I,12) = FZ(4,1)
XOUT(I,13) = FX(5,1)
XOUT(I,14) = FY(5,1)
XOUT(I,15) = FZ(5,1)

```

```

CALL DOSYSWITTIME,TIMSTP,NTRAN,1,XOUT,42,ICHAN,IWH, IDATE,1)
595 CONTINUE

```

ENDFILE 30  
ENDFILE 30  
ENDFILE 31  
ENDFILE 31  
STOP  
END

```

C*****
C
C   SUBROUTINE D0SYSW1STRIM,TIMSTP,NTRAN,NPTS,TDATA,MAXTRN,ICHAN,
C   *      IWHERE,IDATE,IFLAG)
C*****
C
C   PURPOSE THIS ROUTINE IS USED TO OUTPUT DATA TO JOSYS FORMAT
C   TAPES.
C   NOTE THE FORMAT STATEMENTS IN THIS ROUTINE ARE SPECIFIC TO
C   THE NSF PROJECT 1182.08.
C
C   INPUTS  STRIM  START TIME OF THE DATA
C           TIMSTP DELTA TIME FOR EACH STEP
C           NTRAN  NUMBER OF TRANSDUCERS(CHANNELS OF DATA)
C           NPTS   NUMBER OF TIME POINTS
C           TDATA  TRANSDUCER DATA
C           MAXTRN MAXIMUM TRANSDUCERS, USED IN DIMENSIONING TDATA
C           ICHAN  CHANNEL NUMBER OF D0SYS TAPE
C           IWHERE INDICATES WHICH SETS OF TDATA ARE USED AS WHICH
C           TRANSDUCERS. A 0 INDICATES THAT ZEROES ARE TO
C           BE OUTPUT FOR THAT CHANNEL.
C           IDATE  INTEGER DATE(I.E. FEB 3,1981 = 81020)
C           IFLAG  IF.LT.0 OUTPUT HEADER AND DATA
C           IF.EQ.7 OUTPUT HEADER ONLY
C           IF.GT.7 OUTPUT DATA ONLY
C
C   OUTPUTS ALL OUTPUT IS TO THE D0SYS TAPE OR THE LISTING FILE(6).
C*****
C
C   MOD      DATE      BY      REASON
C   1.0      3/81      LJS      ORIGINAL
C*****
C
C   DIMENSION IDATA(NPTS,MAXTRN),IWHERE(NTRAN),OUTDAT(256)
C
C   CHECK FOR WHETHER TO OUTPUT HEADER OR NOT
C
C   IF(IFLAG) 50,50,210
C 50 CONTINUE
C
C   WRITE OUT HEADER
C
C   WRITE(ICHAN,9000) IDATE,DATE,NTRAN,NPTS
C   IF(NTRAN.NE.42) GOTO 100
C
C   IF 42 CHANNELS, MUST BE THE TRANSDUCER INFORMATION
C
C   WRITE(ICHAN,9001)
C   WRITE(ICHAN,9101)
C   WRITE(ICHAN,9201)
C   WRITE(ICHAN,9301)
C   WRITE(ICHAN,9401)
C   WRITE(ICHAN,9501)
C   WRITE(ICHAN,9601)
C   GOTO 200
C
C   IF OTHER THAN 42, MUST BE FORCES.
C
C 100 CONTINUE
C   N=NTRAN

```



C-15

```
*      13HRK2010 N/MM2,/  
*      13HBIEGESPANNUNG,/  
*      12HRK2210 GRAD,/  
*      21HWINKEL DER BIEGEACHSE,/  
*      13HRK3010 N/MM2,/  
*      16HTORSIONSSPANNUNG,/  
*      13HRK4110 N/MM2,/  
*      35HTANGENTIALSPANNUNG DURCH INNENDRUCK,/  
*      13HRK5010 N/MM2,/  
*      18HYVERGLEICHSSPANNUNG)  
9301 FORMAT( 13HRK1011 N/MM2,/  
*      11HZUGSPANNUNG,/  
*      13HRK2011 N/MM2,/  
*      13HBIEGESPANNUNG,/  
*      12HRK2211 GRAD,/  
*      21HWINKEL DER BIEGEACHSE,/  
*      13HRK3011 N/MM2,/  
*      16HTORSIONSSPANNUNG,/  
*      13HRK4111 N/MM2,/  
*      35HTANGENTIALSPANNUNG DURCH INNENDRUCK,/  
*      13HRK5011 N/MM2,/  
*      18HYVERGLEICHSSPANNUNG)  
9401 FORMAT( 13HRK1012 N/MM2,/  
*      11HZUGSPANNUNG,/  
*      13HRK2012 N/MM2,/  
*      13HBIEGESPANNUNG,/  
*      12HRK2212 GRAD,/  
*      21HWINKEL DER BIEGEACHSE,/  
*      13HRK3012 N/MM2,/  
*      16HTORSIONSSPANNUNG,/  
*      13HRK4112 N/MM2,/  
*      35HTANGENTIALSPANNUNG DURCH INNENDRUCK,/  
*      13HRK5012 N/MM2,/  
*      18HYVERGLEICHSSPANNUNG)  
9501 FORMAT( 13HRK1013 N/MM2,/  
*      11HZUGSPANNUNG,/  
*      13HRK2013 N/MM2,/  
*      13HBIEGESPANNUNG,/  
*      13HRK2213 N/MM2,/  
*      21HWINKEL DER BIEGEACHSE,/  
*      13HRK3013 N/MM2,/  
*      16HTORSIONSSPANNUNG,/  
*      13HRK4113 N/MM2,/  
*      35HTANGENTIALSPANNUNG DURCH INNENDRUCK,/  
*      13HRK5013 N/MM2,/  
*      18HYVERGLEICHSSPANNUNG)  
9601 FORMAT( 13HRK1014 N/MM2,/  
*      11HZUGSPANNUNG,/  
*      13HRK2014 N/MM2,/  
*      13HBIEGESPANNUNG,/  
*      13HRK2214 N/MM2,/  
*      21HWINKEL DER BIEGEACHSE,/  
*      13HRK3014 N/MM2,/  
*      16HTORSIONSSPANNUNG,/  
*      13HRK4114 N/MM2,/  
*      35HTANGENTIALSPANNUNG DURCH INNENDRUCK,/  
*      13HRK5014 N/MM2,/  
*      18HYVERGLEICHSSPANNUNG)  
9002 FORMAT( 5HRFOOD, I1, 3H N, 19X, 3HLOZ, 10X, 6H....., / )  
9003 FORMAT( 4HRFOO, I2, 3H N, 19X, 3HLOZ, 10X, 6H....., / )  
9004 FORMAT( 3HRFO, I1, 3H N, 19X, 3HLOZ, 10X, 6H....., / )  
9005 FORMAT( 6E12, 5, 8X )  
END
```



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