

SEISMIC AND STRESS ANALYSIS OF THE LACBWR
HIGH PRESSURE CORE SPRAY
DISCHARGE LINE PIPING SYSTEM

Prepared Under NES Project 5101 for
DAIRYLAND POWER COOPERATIVE

NUCLEAR ENERGY SERVICES, INC.
Danbury, Connecticut 06810

LPDR

Prepared by: A. Obligado

7810100011

I. Husain

Approved by: W. M. ...
Project Manager

PDR ADOCK 050-409 P 780926

A. H. Yc
V. P. Engineer

Date: 5/10/77

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1. SUMMARY

This report, prepared for Dairyland Power Cooperative, presents the results of seismic and stress analyses of the High Pressure Core Spray (HPCS) piping system discharge line for the LACBWR Nuclear Power Station. The seismic and stress analyses are performed in accordance with the design requirements for Class 1 piping components of the ASME Boiler and Pressure Vessel Code, Section III, Division 1, "Nuclear Power Plant Components", 1974. By providing seismic restraints at critical locations of the HPCS discharge line, the stresses in the piping due to a seismic event can be reduced to acceptable values. That is, the stresses due to seismic, deadweight, pressure and thermal expansion loadings, combined according to the ASME Code rules for Class 1 components would satisfy the design requirements given in the Code with the addition of seismic restraints. The detail stress analysis also indicates 2900 permissible HPCS operating cycles.

2. INTRODUCTION

In response to AEC/DL's request to review the effects of an earthquake event on the LaCrosse Boiling Water Reactor, Dairyland Power Cooperative requested Gulf United Nuclear Fuels Corporation to evaluate the adequacy of the major structures and equipment to withstand seismic loadings. The seismic study performed by Gulf United (GU) Nuclear Fuels Corporation (Reference 1) included an analysis of the main steam line which indicated that high stresses would be generated in the main steam line during a seismic event. It was also evident from these analyses that the LACBWR piping systems, in general, were not designed to accommodate horizontal accelerations, the primary earthquake induced loading condition. Anticipating the possibility of a seismically induced loss of coolant accident, it was, therefore, concluded that analyses of the major Class 1 piping systems should be performed to evaluate their structural integrity.

This report presents the seismic and stress analysis for the discharge line of the High Pressure Core Spray (HPCS) System. (The HPCS suction line analyses is presented in a separate report, NES 81A0090). The High Pressure Core Spray System is the principal emergency core cooling system. In order to verify that the seismic stresses in the HPCS discharge line are acceptable, it is necessary to show that the combined stresses in the piping system are within ASME Boiler and Pressure Vessel Code allowable values for Class 1 Components. This requires that the seismic stresses be combined with the stresses due to deadweight, pressure and thermal loadings in accordance with the ASME Code Section III, Subsection NB rules (Reference 2).

For the static and dynamic analysis, the High Pressure Core Spray discharge line, including a portion of the sodium pentaborate discharge line has been mathematically modeled as a finite element model. Seismic restraints were provided at suitable locations in order to reduce the seismic stresses in this piping system and to isolate it for analytical purposes from the effects of the HPCS suction line and the sodium pentaborate discharge line to the 16 inch forced circulation discharge header. The static response of the HPCS Discharge Line to the dead weight, thermal expansion and anchor movement loadings have been calculated using direct stiffness displacement methods of structural analysis. The response of the HPCS Discharge Line to seismic loadings have been determined using response spectrum modal superposition methods. Stresses due to various loadings have been calculated and combined in accordance with the ASME Code Section III, Subsection NB rules.

Initial stress calculations, using the conservative stress intensification factors of the ASME Code for the socket weld coupling/reactor nozzle region, resulted in an excessive peak stress value and therefore a very low number of permissible HPCS operating cycles (100 cycles). The socket weld coupling/reactor nozzle region was re-analyzed by means of a detailed finite-

element model using the ANSYS computer code. The transient thermal loadings produced by the HPCS initiation were determined using the same basic model developed for stress analysis and using the LION computer program.

Section 3.0 of this report describes the High Pressure Core Spray (HPCS) Discharge Line considered in the analysis. The loading criteria, design criteria and analytical methods used in the analyses are given in Section 4.0, 5.0 and 6.0 respectively. The results of the analyses are discussed in Section 7.0. The conclusions are summarized in Section 8.0.

3. DESCRIPTION OF PIPING SYSTEM

The High Pressure Core Spray (HPCS) System of the LACBWR power plant is designed to provide an emergency coolant spray to the reactor core in the event that reactor water level drops accidentally. This is done by either direct gravity feed of water from an overhead storage tank to the core spray header under low reactor pressure conditions, or by means of high pressure water injection under high reactor pressure conditions.

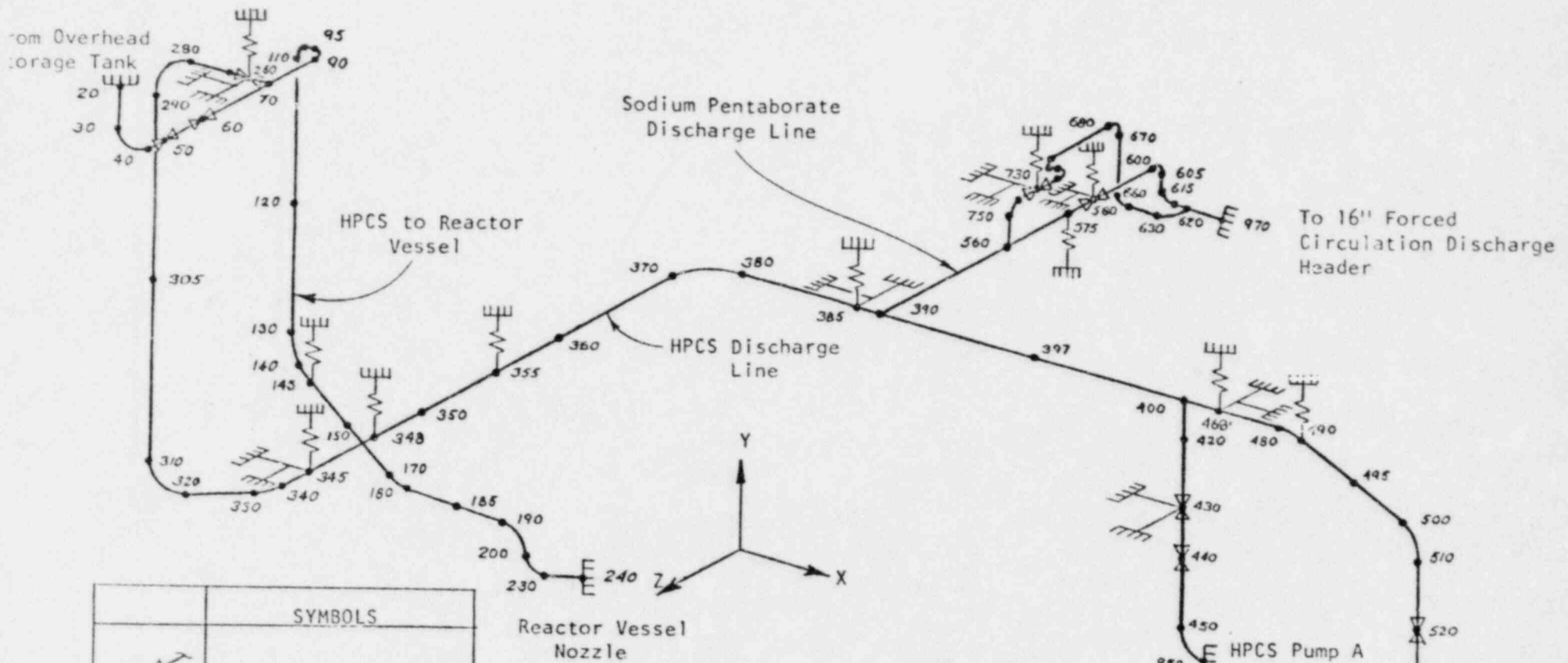
In order to simplify the piping system analysis, the long and complex HPCS piping system was divided into two sections: the first consisting generally of the suction piping which runs from the overhead storage tank to the high pressure core spray pumps and the second consisting of the discharge piping which runs from the high pressure core spray pumps to the core spray header inlet. The HPCS suction piping analysis is presented in a separate report. The subject analysis of this report is, therefore, the HPCS discharge line.




The HPCS discharge line consists of stainless steel pipe line leading from the two high pressure core spray pumps to the core spray header inside the reactor vessel. The pumps are used for core spray when the reactor remains pressurized, as in the case of a small leak below the core. When the reactor and containment building pressures are equalized, as after a major system leak or rupture, a low pressure supply line bypassing the emergency core spray pumps allows water to flow directly from the overhead storage tank (or service water line) to the core spray header. The high pressure core spray pumps are also used in the boron injection system. Redundant control valves are provided for this purpose in the core spray pumps suction and discharge lines.

Rigid anchors located at points of expected large seismic deflections serve the purpose of isolating the discharge lines from the interconnecting piping systems. Figure 3.1 shows the routing of the discharge line and the extent of suction line and sodium pentaborate lines considered in the subject analysis.

The governing design specification used in the analysis of the HPCS piping system is given in Reference 3. The piping arrangement has been taken from the drawings listed in Reference 4. Piping properties have been taken from the information given in the piping specification (Reference 3). The location of piping suspension (hangers, etc.) and their dimensional characteristics were determined from the piping drawings and actual visual inspection and measurements at the LACBWR site. Pipe properties and valve weights are summarized in Tables A-1 and A-II of Appendix A.

FIGURE 3.1
 PIPESD MATHEMATICAL MODEL
 LACBWR HPCS DISCHARGE LINE



SYMBOLS	
	Valve
	Spring Hanger
	Rigid Restraint (Anchor)
Node 50	No Rotations Allowed

NOTE: Restraints at nodes 20, 970 and 50 to be provided.

4. LOADING CRITERIA

The loading conditions which must be taken into account in performing a Class I analysis of a piping system are specified in Subsection NB-3110 of Reference 2. These include dead weight, internal pressure, thermal effects; and earthquake loads. Design, operating, upset, emergency and faulted condition loadings must be considered in the analysis as specified in the stress acceptance criteria (Section 5 of this report). The static and dynamic load cases considered in the analysis are described below and the detail input data are summarized in Table A-III and A-IV of Appendix A.

4.1 Load Case 1 and 2 - Design and Operating Pressures

Piping design pressures are taken from the LACBWR piping specification (Reference 3) and are 100 psig for the piping between node points 20 and 50 and 1400 psig elsewhere in the system.

Operating pressures for the HPCS system are based on the LACBWR Safeguards Report (Reference 5). These are 100 psig up to node point 50, 1340 psig from node 50 to the reactor vessel nozzle and 1400 psig for the remainder of the system.

4.2 Load Case 3 - Dead Weight and Other Sustained Mechanical Loads

The dead weight of the piping system is calculated considering the piping to be insulated and filled with water. Other sustained loads included in the analysis are valve weights, valve operator weights, and the tributary weights from branch piping.

4.3 Load Case 4 - Seismic Anchor Movements

Seismically induced anchor movements for the OBE earthquake were estimated by calculating low frequency displacements from the containment vessel response spectra (from Reference 1) at different elevations.

4.4 Load Case 5 - Thermal Anchor Movements

Thermal expansion or contraction of the reactor vessel during start-up and shutdown results in maximum displacements of the piping system anchor at the reactor vessel nozzle (node point 240).

4.5 Load Cases 6, 7, 10 and 11 - Seismic Loading

A dynamic analysis of the piping system is performed using the response spectrum modal superposition method of analysis (Section 6.4). Two seismic loading events are considered: the safe shutdown earthquake (SSE), and the operating basis earthquake (OBE). The established design criteria

(Reference 6, Regulatory Guide 1.48, May, 1973) for Class 1 analysis specifies that the OBE (or 1/2 SSE) must be considered in conjunction with the normal and upset plant condition while the SSE must be considered in conjunction with the faulted plant condition.

Seismic inertia loading is imposed on the piping system in the form of seismic acceleration spectra which were derived for the LACBWR plant (Reference 1). The horizontal acceleration spectra used for the HPCS discharge line is that corresponding to the subsystem support points on the reactor containment shell at an elevation of 695 feet. The vertical response spectrum for the SSE loading is taken as 2/3 of the horizontal SSE ground response spectrum assuming no amplification of vertical response in the structure. For the Operating Basis Earthquake the vertical piping response spectrum is taken as 1/2 of the SSE vertical response spectrum. Damping values used are 1 percent for the OBE and 2 percent for the SSE.

The horizontal spectra in either the global X- direction or the global Z- direction are applied simultaneously with the vertical spectra in the global Y- direction. Load cases 6 and 7 represent the Operating Basis Earthquake while 10 and 11 represent the SSE earthquake. The applicable response spectra used in the analysis for dynamic load cases are shown in Table A-IV of Appendix A.

4.6 Load Case 8 - Thermal: Maximum Credible Accident

The sudden introduction of cold water from the HPCS system piping into the hot pressure vessel nozzle, due to a LOCA or other low water level condition results in a transient thermal condition in the nozzle region. This temperature transient generates stresses in the pipe due to the large temperature gradients across the pipe wall and due to any material discontinuities present. These thermal loads which are applied at node points 230 and 240 have been calculated by means of a transient thermal analysis with the Lion Computer Code (Appendix E) and are presented in Appendix B. These loads are considered in conjunction with the upset plant condition.

4.7 Load Case 9 - Thermal: Normal Start-Up and Shutdown

During normal start-up and shutdown a temperature change of 344°F is assumed in the piping in the region of the reactor vessel HPCS discharge nozzle.

5. STRESS ACCEPTANCE CRITERIA

The requirements for acceptability of a Class 1 piping system are given in AEC Regulatory Position 1 of Reference 6 and Subsections NB-3600 of Section III of the ASME Boiler and Pressure Vessel Code, Reference 2. Calculated stresses resulting from the design and operating loading conditions given in Subsection NB-3110 and NB-3620 must meet the stress limits of equations 9 through 14 of Subsection NB-3650 of the ASME Code.

5.1 Design Conditions

The primary stress intensity, resulting from the combined effects of the design pressure (Load Case 1) and the resultant moment loading due to loads caused by dead weight (Load Case 3) and the Operating Basis Earthquake (Load Cases 6 and 7), and calculated in accordance with equation 9 of Subsection NB-3652 of the Code must be less than 1.5 times the allowable design stress intensity, S_m , at maximum temperature.

5.2 Normal Conditions

The primary plus secondary stress intensity range resulting from the combined effects of thermal expansion, linear thermal gradient and discontinuity effects (Load Case 8), operating pressure (Load Case 2), anchor movements (Load Case 4 and 5) and earthquake effects (Load Cases 6 or 7), calculated in accordance with equation 10 of the Code must be less than 3 times S_m . In the event that the above requirement is not met the piping product may still be acceptable provided the requirements of a simplified elastic-plastic discontinuity analysis are met. This requirement is met if 1) the nominal expansion stress resulting from thermal expansion and thermal anchor movements calculated in accordance with equation 12 of the Code is less than 3 S_m and 2) if the range of primary plus secondary membrane plus bending stress intensity, resulting from the combined loading of operating pressure, dead weight, one-half the range of the earthquake and thermal discontinuity stresses, calculated according to equation 13 of the code is less than 3 S_m .

The requirements for acceptability under cyclic loading conditions are met by first calculating the peak stress intensity by means of equation 11 of the Code, resulting from the loadings specified for equation 10 plus the loadings resulting from the non-linear portion of the thermal gradient through the wall thickness and then calculating the alternating stress intensity in accordance with equation 14 of the Code. The total number of operating stress cycles must then be less than those determined from the fatigue curves from Appendix J-9 of the Code for the calculated alternating stress intensity in accordance with the requirements of paragraphs NB-3653.4 and NB-3653.5 of the Code.

5.3 Upset Conditions

The requirements for acceptability under upset conditions are the same as for Normal Conditions.

5.4 Emergency Conditions

The requirement for acceptability under emergency conditions (not specified in this analysis) is that the primary stress intensity, as calculated by equation 9 of the Code, must be less than $2.25 S_m$.

5.5 Faulted Conditions

Under faulted conditions the primary stress intensity resulting from the combined effects of design pressure (Load Case 1), dead weight (Load Case 3) and the vibratory motion of the full Safe Shutdown Earth uake (Load Cases 10 and 11) as calculated by equation 9 of the Code must be less than $3 S_m$.

6. ANALYTICAL METHODS

6.1 Mathematical Model

In order to perform static, dynamic and stress analyses, the continuous piping system is mathematically modeled as an assembly of elastic structural elements interconnected at discrete nodal points (Figure 3.1). Nodal points are located at all points of interest in the piping system such as elbows, valves, anchorages, hangers, tee intersections, load points, all structural and material discontinuities, etc. This three dimensional multidegree-of-freedom model of the piping system is attached to the "ground" (structure) by means of rigid hangers, support springs, hydraulic snubbers and anchors. Stiffness characteristics of structural elements are related to the moment of inertia and the axial and effective shear area of the pipe cross section. The stiffness characteristics of the elbows and tee connections are modified to account for local deformation by using the flexibility factors given in the ASME Code (Reference 2).

For the seismic analysis the distributed mass of the piping system is lumped at the system nodal points. Masses are lumped so that the lumped mass, multidegree-of-freedom model represents the dynamic characteristics of the piping system. In order to reduce the number of dynamic degrees-of-freedom, only translational degrees-of-freedom are considered at each mass point (the masses associated with the rotational degrees-of-freedom are set to zero). This assumption has been shown to be completely satisfactory for accurate analysis of seismic response. Special items such as valves and actuators are modeled by lumping their masses at an appropriate offset from the center-line of the piping system.

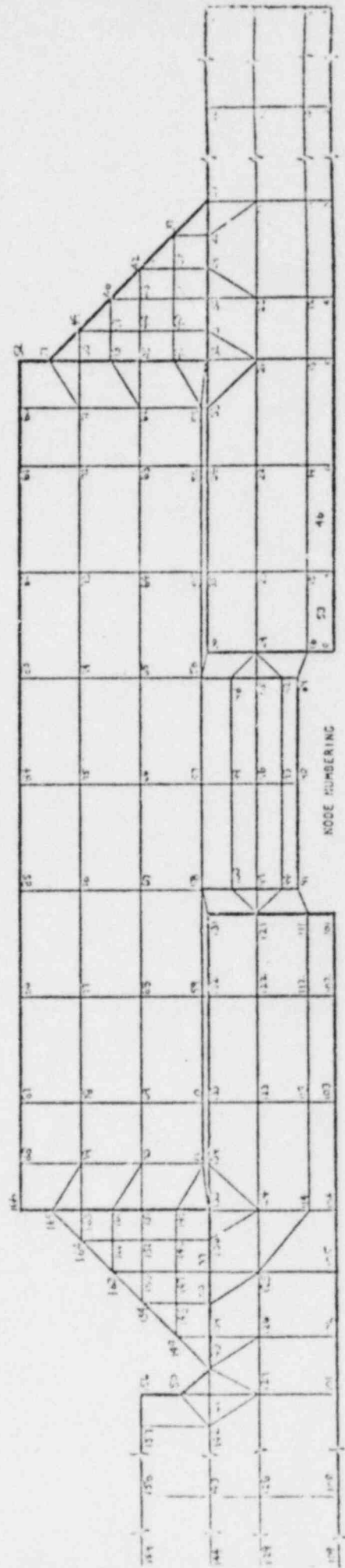
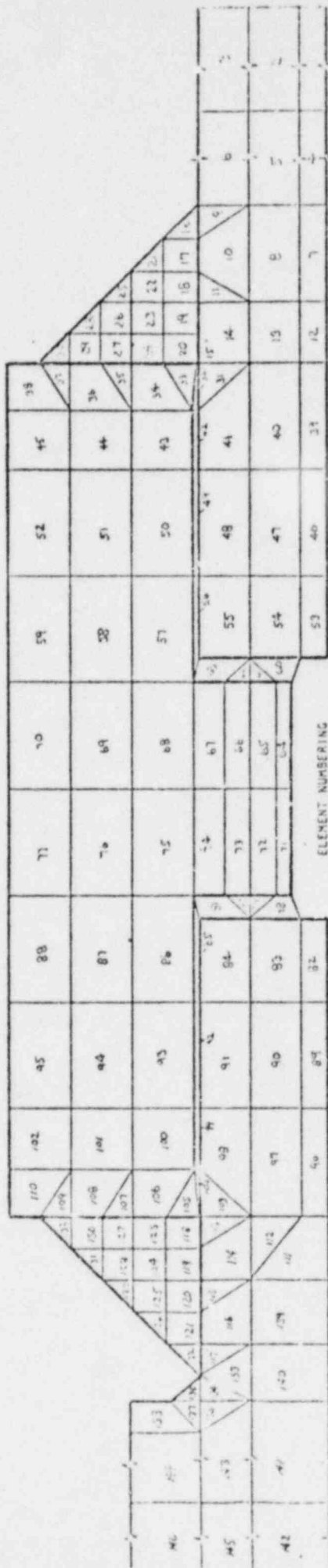
Figures 6.1 and 6.2, show the schematic sketches of the ANSYS finite element model of the socket weld coupling/reactor nozzle region for determining the peak stresses resulting from the thermal transient produced by HPCS initiation. The triangular and quadrilateral axisymmetric ring finite element model sufficiently represents the reactor vessel nozzle and its geometrical variations, the Inconel transition piece, the Inconel transition piece weld to the stainless steel (SS) socket, the SS socket to SS piping weld, and the SS piping. The same basic model with larger element sizes has been used for the transient thermal analysis using the LION computer program. Appendix E gives the details of the transient thermal analysis. The thermal analysis provides the transient temperature history for each finite element, which is used as an input thermal loading to the ANSYS analysis.

6.2 Static Load Analysis

The static load analysis involves the application of the following loading conditions and their combinations:

FIGURE 6.2

HPCS DISCHARGE NOZZLE SOCKET WELD
MODEL ELEMENT AND NODE NUMBERING



- . Design Pressure
- . Gravity Loading (dead weight) and Sustained Mechanical Loads
- . Support Displacement
- . Thermal Expansion

For the pressure loadings, the hoop and longitudinal stresses in the affected piping are calculated using the formulae given in the Code (see Section 6.5).

For the dead weight, support displacement, or thermal expansion loading conditions the following equations of equilibrium written in matrix form are solved:

$$KU = P \quad (1)$$

where:

- K = System stiffness matrix
- U = Nodal point displacement vector
- P = External forces, dead weight or equivalent thermal load vector.

The system stiffness matrix is obtained from element stiffness matrices using direct stiffness methods. The unknown nodal displacements U are obtained as follows:

$$U = K^{-1} \times P \quad (2)$$

The inversion of the stiffness matrix is performed using the Gauss-Seidel technique.

From the nodal displacements U, the member internal forces are determined using the member stiffness matrix. Finally the member internal forces are used in calculating the stresses.

6.3 Eigenvalue Analysis

The eigenvalues (natural frequencies) and the eigenvectors (mode shapes) for each of the natural modes of vibration are calculated by solving the following frequency equation:

$$[K - \omega_n^2 M] \{\phi_n\} = \{0\} \quad (3)$$

where:

- ω_n = Natural angular frequency for the n^{th} mode
- M = System mass matrix
- ϕ_n = Mode shape vector for the n^{th} mode
- 0 = Null vector

The eigenvalue/eigenvector extraction is performed using the Householder-QR technique.

6.4 Dynamic (Seismic) Load Analysis

Considering only translational degrees of freedom and assuming viscous (velocity proportional) form of damping, the equation of motion in matrix form can be expressed as follows:

$$M(\ddot{U}_t + \ddot{U}_{gt}) + C\dot{U}_t + KU_t = 0 \quad (4)$$

where:

\ddot{U}_t = Relative acceleration time history vector

\ddot{U}_{gt} = Ground acceleration time history vector

C = Damping matrix

\dot{U}_t = Velocity time history vector

U_t = Relative displacement time history vector

Rearranging equation (4)

$$M\ddot{U}_t + C\dot{U}_t + KU_t = -M\ddot{U}_{gt} = P_{eff} \quad (5)$$

To uncouple equation (5), assume

$$U = \phi Y_t$$

where:

ϕ = Characteristic free vibration mode shapes matrix

Y_t = Generalized coordinate displacement time history vector

Applying the above coordinate transformation and multiplying equation (5) by the transpose of ϕ and using orthogonality conditions, the following uncoupled equations of motion are obtained:

$$\ddot{Y}_{nt} + 2\omega_n \lambda_n \dot{Y}_{nt} + \omega_n^2 Y_{nt} = M_n^{*-1} R_n \ddot{U}_{gt} \quad (6)$$

where:

Y_{nt} = Generalized displacement coordinate time history for n^{th} mode.

λ_n = Damping ratio for the n^{th} node expressed as percent of critical damping

M_n^* = Generalized mass for the n^{th} node

$$= \phi_n^T M \phi_n = M_i \phi_{in}^2$$

The mode shape ϕ_n is normalized such that $M_{ii}^* = 1$

R_n = Participation factor for the n^{th} mode

$$= \phi_n^T M I = \sum M_i \phi_{in}$$

I = Column vector whose elements are generally unity

The solution for the differential equation (6) is given by the Duhamel Integral:

$$Y_{nt} = \frac{R_n}{M_n^* \omega_n} \int_0^t \ddot{U}_{gt} e^{-\lambda_n \omega_n (t-T)} \sin \omega_n (t-T) dT$$

Using the response spectrum method of analysis, the maximum value of the generalized response for each mode is given by:

$$\dot{Y}_n \text{ max} = \frac{R_n S_{an}}{M_n^*}$$

where:

$\dot{Y}_n \text{ max}$ = Maximum generalized coordinate acceleration response for the n^{th} mode.

S_{an} = Spectral acceleration value for the n^{th} mode (from the applicable response spectrum curve)

From the maximum generalized coordinate response, the maximum acceleration ($\ddot{U}_n \text{ max}$) and maximum inertia forces ($F_n \text{ max}$) at each mass point are given by:

$$\ddot{U}_n \text{ max} = \dot{Y}_n \text{ max} \phi_{in}$$

$$F_n \text{ max} = M_n \ddot{U}_n \text{ max}$$

The inertial forces ($F_n \text{ max}$) for each of the system natural modes are applied as external static forces, and the piping system response (displacements, member internal forces and stresses) are calculated using the procedure described in Section 4.2. Total system response is then obtained by combining the individual modal response values by the square-root of the sum of the squares method; lower modes having large contribution to the response (all modes having natural frequency under 30 cycles per second) are considered and high modes with negligible participation are neglected.

6.5 Stress Analysis

The design requirements of Section III of the ASME Boiler and Pressure Vessel Code, Reference 2 (henceforth referred to as the "Code") for Class I piping systems are satisfied when the calculated stresses in the piping system due to thermal expansion, weight, and other sustained and mechanical loads are combined in accordance with, and meet the limitations of Subsection NB-3600 of the Code. These requirements are described below.

5.5.1 Pressure Design Check

The minimum pipe wall thickness requirements for the design pressure are met by satisfying equation (1).

$$t_m = \frac{PD_o}{2(S_m + YP)} \quad (1)$$

where

t_m = the minimum required wall thickness, in.

P = internal design pressure, psi.

D_o = outside diameter of pipe, in.

S_m = maximum allowable stress in the material at the design temperature from Tables I-1.0 of the Code, psi.

Y = 0.4

6.5.2 Consideration of Design Conditions

The primary stress intensity limit is satisfied by meeting the requirements of equation (9)

$$B_1 \frac{PD_o}{2t} + B_2 \frac{D_o}{2I} M_i \leq 1.55 S_m \quad (9)$$

where:

B_1, B_2 = primary stress indices for the specific product under investigation (From Subsection NB-3680 of the Code)

P = design pressure, psi

D_o = outside diameter of pipe, in.

t = nominal wall thickness of component

I = moment of inertia, in.⁴

M_i = resultant moment loading due to loads caused by (1) weight, (2) earthquake, considering only one-half the range of the earthquake and excluding the effects of anchor displacements due to earthquake, and (3) other sustained design mechanical loads.

S_m = allowable design stress intensity value, psi

6.5.3 Consideration of Normal Conditions

Protection against fatigue failure is provided for by means of one of the two analyses dependent on whether the structure is subjected to elastic cycling or plastic cycling. The criterion for establishing whether the structure cycles

in the elastic range or the plastic range is set forth in equation (10) of the ASME Code. Compliance with Equation (10) assures that, after a few cycles of load application, the maximum stress will remain within the range of tensile and compressive yield strengths, i.e. within the elastic range. If this criterion is met, the fatigue evaluation is based on purely elastic behavior. If the criterion is not met, an elastic-plastic discontinuity analysis must be made.

Determination of Primary Plus Secondary Stress Intensity Range Limitations

This calculation is based upon the effect of changes which occur in mechanical or thermal loadings which take place as the system goes from one load set, such as pressure, temperature, moment, and force loading, to any other load set which follows it in time. It is the range of pressure, temperature, moment, between two load sets which is to be used in the calculations.

The primary plus secondary stress intensity range limitations are satisfied by meeting the requirements of equation (10).

$$S_n = C_1 \frac{P_0 D_0}{2t} + C_2 \frac{D_0}{2l} M_i + \frac{1}{2(1-\nu)} E \alpha |\Delta T_1| + C_3 E_{ab} |\alpha_a T_a - \alpha_b T_b| \leq 3S_m$$

where:

C_1, C_2, C_3 = secondary stress indices for the specific component under investigation (NB-3680)

D_0, t, l, S_m = are as defined for Equation (9)

M_i = range of moment loading due to (1) thermal expansion (2) anchor movements from any cause, (3) earthquake effects, and (4) other mechanical loads.

$|\Delta T_1|$ = range of absolute value (without regard to sign) of the temperature difference between the temperature of the outside surface (T_o) and the temperature of the inside surface (T_i) of the piping product assuming moment generating equivalent linear temperature distribution.

$T_a(T_b)$ = range of average temperature on side a(b) of gross structural discontinuity or material discontinuity.

$\alpha_a(\alpha_b)$ = coefficient of thermal expansion on side a(b) of a gross structural discontinuity or material discontinuity at room temperature.

E_{ab} = average modulus of elasticity of the two sides of a gross structural discontinuity or material discontinuity at room temperature, psi.

$E\alpha$ = modulus of elasticity (E) times the mean coefficient of thermal expansion (α) both at room temperature, psi.

ν = poisson's ratio = 0.3

P_0 = range of operating pressure, psi

Determination of Peak Stress Intensity

The peak stress intensity is calculated by means of equation (11) of the Code for every pair of load sets. This is the maximum stress intensity at a point including any local structural discontinuity (or notch) effects and any local thermal stresses.

$$S_p = K_1 C_1 \frac{P_0 D_o \alpha}{2t} + K_2 C_2 \frac{D_o}{2l} M_i + \frac{1}{2(1-\nu)} K_3 E \alpha |\Delta T_1| + K_3 C_3 E_{ab} |\alpha_a T_a - \alpha_b T_b| + \frac{1}{1-\nu} E \alpha |\Delta T_2| \quad (11)$$

where:

K_b, K_2, K_3 = local stress indices for the specific component under investigation (NB-3680)

$E\alpha$ = same as in Equation (10)

ΔT_2 = range of absolute value (without regard to sign) for that portion of the nonlinear thermal gradient through the wall thickness not included in ΔT_2 of Equation 10 °F below.

For a quantitative definition of $|\Delta T|$ and $|\Delta T_2|$, see NB-3653.2(b) of the Code. All other terms are as defined in Equation (10).

The peak stress, S_p , is used to calculate the alternating stress intensity, S_{alt} , for the fatigue evaluation

Simplified Elastic-Plastic Discontinuity Analysis

If the primary plus secondary stress intensity requirements of equation (10) are not met, the fatigue evaluation must include the effects of plastic cycling by means of the simplified elastic-plastic discontinuity analysis as described below. Only those pairs of load sets not satisfying equation (10) need be considered. Equation (12) imposes a limitation of $3 S_m$ on the magnitude of the thermal expansion stress to prevent possible collapse due to the development of a hinge moment.

$$S_e = C_2 \frac{D_o D_i}{2t} M_i^* \leq 3S_m \quad (12)$$

where:

S_e = nominal value of expansion stress

M_i^* = same as M_i in Equation (10) except it includes only moments due to thermal expansion and thermal anchor movements

The range of primary plus secondary membrane plus bending stress intensity, excluding thermal bending and thermal expansion stresses shall be $\leq 3S_m$. This requirement is satisfied by meeting Equation (13) below.

$$C_1 \frac{P_o D_o}{2t} + C_2 \frac{D_o M_i}{2t} + C_3 E_{ab} |d_b T_a - d_b T_b| \leq 3S_m \quad (13)$$

Fatigue Evaluation

If the conditions of equation (10) are met, or alternatively the conditions of equations (12) and (13) are met, the value of the alternating stress intensity, S_{alt} , is calculated by equation (14) of the Code.

$$S_{alt} = K_e \frac{S_p}{2} \quad (14)$$

where:

S_{alt} = alternating stress intensity

S_p = peak stress intensity value calculated by Equation (11)

K_e = 1.0 for $S_n \leq 3S_m$ (i.e. when equation 10 is satisfied)

$$= 1.0 + \frac{(1-n)}{n(m-1)} \left[\frac{S_n}{3S_m} - 1 \right] \text{ for } 3S_m < S_n < 3M S_m$$

$$= \frac{1}{n} \text{ for } S_n \geq 3M S_m$$

S_n = primary plus secondary stress intensity value calculated in Equation (10)

m and n = material parameters given in NB-3228.3 (b) of the Code

The number of allowable cycles, M , for a given stress cycle is then determined from applicable design fatigue curves given in Appendix I-9.0 of the Code for the calculated values of S_{alt} . If more than one operational cycle is being considered which produces significant fluctuating stresses, a cumulative usage factor must be determined in accordance with Subsection NB-3222.4 of the Code.

The cumulative usage factor, U, is defined as:

$$U = \sum U_i = \sum \frac{n_i}{N_i}$$

where:

- U_i = usage factor for each type of stress cycle, i
- n_i = specified number of times a given stress cycle, i, will be repeated during the life of the component
- N_i = allowable number of repetitions for a given stress cycle, i, from table I-90 of the Code.

The cumulative usage factor, U, must not exceed 1.0.

Stress Range Calculations

The stress range evaluation is carried out by means of one or both of two analyses. The first analysis is a maximum stress range calculation in which the maximum range of stresses from each load set pair is used to form a "worst load case" which is assumed to occur over the total number of system cycles. If this conservative check results in an allowable number of cycles not exceeding the total number of system cycles, then no further analysis is required for the component. If the maximum stress range check fails, an individual stress range calculation can be made to establish component acceptability. In this evaluation the stress ranges for individual load sets are calculated, pair by pair, in such a manner as to maximize stress ranges and the cumulative usage factor, which must be less than 1.0.

6.5.4 Consideration of Upset Conditions

The procedure and stress limits for evaluating upset conditions are the same as for operating conditions.

6.5.5 Consideration of Emergency Conditions

The primary stress intensity requirements of equation (9) above (Section 5.2) must be met using a stress limit of $2.25 S_m$.

6.5.6 Consideration of Faulted Conditions

The primary stress intensity requirements of equation (9) above (Section 6.6.2) for the combined loading effects of system design pressure, deadweight and the vibratory motion of the Safe Shutdown Earthquake must be met using a stress limit of $3S_m$.

7. DISCUSSION OF RESULTS

The results of the HPCS discharge line piping analysis are based on the assumption that additional restraints will be provided at node points 20, 970 and 50 as indicated in Figure 3.1. These restraints may be rigid anchors due to low thermal expansion effects in the HPCS line.

The natural frequencies of the lower modes of vibration of the piping system, up to 35 cycles per second, are given in Table 7.1 and indicate a generally flexible (low frequency) system. System deflections resulting from the various load cases are presented in appendix B, Table B-1. The maximum deflection due to dead weight is 0.076 inch at node point 170, while the maximum deflection due to the SSE seismic inertia loading is 0.680 inch at node point 140. For a flexible piping system these deflections are within acceptable values.

Table B-11 of Appendix B gives the elastic support reaction forces resulting from the various load cases. These forces should be used in the design of the additional system restraints.

The results of the detailed stress analysis in accordance with the requirements of Subsection NB-3650 of the ASME Code for Class 1 piping systems are summarized in Figures 7.1 through 7.12. Significant stresses calculated by the PIPESD computer analysis using Code equations 9 through 14 for the various load combinations are indicated in these figures and are compared to the Code allowable values.

The peak stress intensity at the pressure vessel discharge nozzle socket weld (node point 240) resulting from the thermal transient (See Appendix B) and material and structural discontinuities has been calculated by means of the detailed ANSYS finite element analysis. Table 7.2 presents the results of this analysis. Table 7.2 shows that a peak thermal stress of 93.5 ksi results seven seconds after initiation of the thermal transient. This peak thermal stress is combined with the PIPESD calculated pressure and seismic stresses at node 240 (See Appendix C) to determine a peak stress intensity range, S_p of 161.7 ksi and an alternating stress intensity, S_{alt} , of 80.8 ksi (Figure 7.10). The maximum allowable number of full stress cycles for the nozzle socket weld is then 2900.

TABLE 7.1
 NATURAL FREQUENCIES OF VIBRATION
 HPCS DISCHARGE LINE

<u>MODE</u>	<u>FREQUENCY (CPS)</u>	<u>MODE</u>	<u>FREQUENCY (CPS)</u>
1	3.23	14	18.84
2	5.64	15	20.76
3	6.68	16	22.04
4	7.85	17	22.27
5	9.03	18	23.42
6	9.40	19	25.20
7	11.61	20	26.31
8	11.81	21	26.62
9	12.15	22	26.77
10	13.37	23	28.39
11	15.26	24	30.02
12	16.84	25	30.63
13	18.07	26	32.98
		27	33.85

TABLE 7.2

RESULTS OF ANSYS STRESS ANALYSIS SUMMARY OF PEAK STRESSES VS TIME IN SOCKET WELD REGION

ELEMENT NO.	Stress, ksi														
	Time, Seconds														
	0	0.25	0.50	0.75	1.0	2.0	3.0	4.0	5.0	6.0	7.0	8.0	9.0	10.0	15.0
1	0.2	4.1	12.3	20.9	23.9	26.1	21.2	16.7	13.1	10.2	8.0	6.0	5.5	4.1	2.5
4	0.6	7.0	18.5	31.4	28.6	47.2	43.6	39.4	35.0	32.3	29.8	27.1	25.6	24.0	19.1
7	2.0	8.8	20.1	33.5	40.9	51.7	49.2	47.4	45.4	43.9	41.9	40.3	39.7	38.0	33.2
12	1.2	7.4	19.7	34.8	43.0	58.5	57.6	55.0	52.2	49.6	47.5	45.8	44.1	42.4	37.0
14	1.5	2.5	5.7	10.3	14.6	16.2	15.0	12.3	11.4	10.8	10.4	10.6	10.3	10.3	8.8
16	4.2	4.1	8.1	16.3	38.0	41.8	46.1	46.0	49.6	48.3	49.6	47.4	44.9	44.1	38.9
20	1.0	2.1	5.0	9.6	15.7	28.7	35.4	41.1	42.2	44.5	44.4	47.4	49.6	47.1	47.5
24	1.6	1.9	4.0	7.7	12.7	24.1	29.1	33.0	37.3	36.6	40.2	38.8	36.6	43.9	36.8
31	2.6	1.6	3.6	7.4	12.5	10.1	14.2	19.1	24.2	27.3	30.4	32.3	33.5	34.7	34.9
33	4.2	4.2	5.7	10.1	16.3	32.9	44.6	52.9	56.6	60.8	61.7	64.3	65.7	63.4	61.4
39	0.6	8.7	21.5	36.6	46.3	63.1	65.1	65.1	64.3	63.1	62.2	61.3	60.2	59.7	53.0
46	1.5	10.1	23.0	38.9	49.1	66.5	70.2	72.0	72.8	72.8	72.5	72.2	71.4	70.2	64.5
47	2.0	1.6	3.2	7.5	11.6	28.1	37.0	43.1	48.8	51.9	54.5	55.9	56.8	57.0	54.8
53	3.1	11.0	22.5	36.2	46.2	60.2	63.7	66.3	67.4	67.9	67.8	67.7	67.1	66.2	61.2
54	3.9	3.8	5.6	9.5	13.7	27.2	35.0	41.0	46.5	49.6	52.2	53.6	54.5	54.8	52.7
64	7.7	13.4	22.5	34.6	46.1	63.3	66.3	67.5	67.0	66.0	65.2	63.9	62.7	61.4	56.1
65	6.9	6.8	8.2	11.2	17.4	34.6	40.1	43.5	44.8	45.2	45.0	44.8	44.2	43.7	40.9
71	10.4	15.1	24.5	34.8	45.7	63.2	65.6	67.6	67.3	66.4	65.6	64.4	63.3	62.1	57.1

TABLE 7.2 (continued)

RESULTS OF ANSYS STRESS ANALYSIS SUMMARY OF PEAK STRESSES VS TIME IN SOCKET WELD REGION

ELEMENT NO.	Stress, ksi														
	0	0.25	0.50	0.75	1.0	2.0	3.0	4.0	5.0	6.0	7.0	8.0	9.0	10.0	15.0
72	8.8	7.3	9.5	12.9	17.5	27.3	40.9	44.3	45.7	46.1	46.0	45.9	45.4	44.9	42.4
82	22.3	28.2	38.7	51.5	60.5	74.7	77.6	79.1	79.9	80.3	80.2	79.8	79.0	78.3	73.2
83	21.2	20.8	23.4	25.0	31.5	41.5	46.1	55.2	58.8	61.5	63.8	64.8	65.6	65.8	63.6
89	19.9	26.0	39.4	54.3	65.5	84.4	86.6	91.3	92.5	93.4	93.5	93.5	92.9	92.4	87.5
90	18.0	17.2	21.5	24.3	32.6	46.4	52.8	63.6	68.1	71.5	74.1	75.1	76.6	77.1	75.3
96	13.4	19.2	31.9	45.6	56.0	73.6	76.4	79.9	81.0	82.1	82.0	82.3	82.0	81.9	78.4
97	11.8	9.8	13.6	16.4	21.1	33.8	40.5	46.4	50.0	52.6	54.5	55.9	56.7	57.6	57.0
100	16.8	17.9	18.2	19.7	22.1	29.2	38.4	40.9	44.3	47.0	48.8	50.3	51.2	51.4	50.8
103	14.3	51.1	13.7	14.3	16.9	25.3	29.2	33.9	36.8	37.9	39.3	40.7	41.5	42.1	42.3
105	19.6	23.8	21.9	24.5	29.9	44.1	59.1	67.5	73.7	78.5	81.3	83.6	84.3	85.4	79.4
111	8.6	11.7	16.2	22.0	28.9	38.9	45.9	47.6	49.0	51.5	51.4	52.3	52.4	52.8	51.1
112	6.6	7.0	9.9	15.1	19.8	35.1	45.2	47.6	50.9	53.3	54.4	55.4	55.4	55.7	54.0
113	17.8	20.8	19.9	22.5	26.4	37.2	50.7	57.2	63.6	67.6	71.9	73.3	74.1	74.7	72.2
117	7.0	7.1	9.1	10.8	7.1	16.9	21.5	23.0	25.8	27.1	28.8	27.7	28.4	29.4	29.8
118	19.6	23.2	21.9	24.2	24.4	34.7	42.6	47.1	50.8	54.1	55.9	56.7	57.5	58.4	58.1
122	17.4	16.7	20.8	23.9	32.7	38.2	47.2	51.6	55.8	60.9	62.7	65.3	66.7	68.9	68.8
123	21.2	25.1	23.1	25.0	27.0	33.8	40.8	44.3	48.1	50.8	51.7	53.4	53.7	53.8	54.0
127	20.5	23.8	21.6	22.7	22.8	28.0	31.4	34.3	35.9	38.2	39.1	40.3	40.8	41.8	42.0

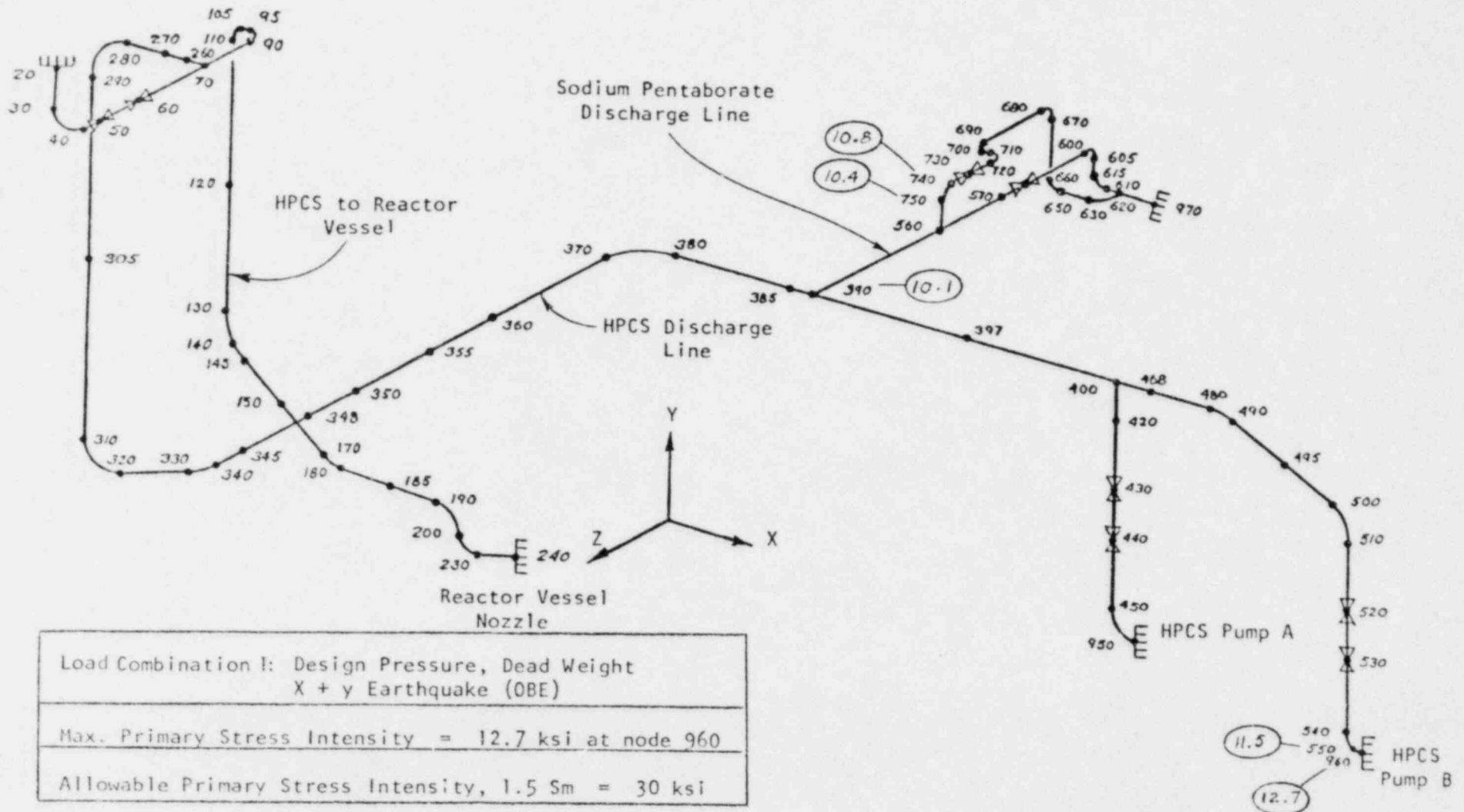
TABLE 7.2 (continued)

RESULTS OF ANSYS STRESS ANALYSIS SUMMARY OF PEAK STRESSES VS TIME IN SOCKET WELD REGION

ELEMENT NO.	Stress, ksi														
	Time, Seconds														
	0	0.25	0.50	0.75	1.0	2.0	3.0	4.0	5.0	6.0	7.0	8.0	9.0	10.0	15.0
130	15.7	18.8	16.0	16.3	16.5	17.3	18.1	19.4	20.1	21.6	21.6	21.9	21.6	22.2	21.8
132	28.3	35.4	29.0	29.6	30.5	31.6	31.9	33.0	33.5	34.5	35.4	35.6	36.0	36.5	36.6
133	4.6	3.9	6.0	7.3	8.0	12.7	16.1	16.8	17.1	16.5	17.9	20.3	20.5	20.6	20.5
134	12.4	11.6	14.7	16.9	16.9	26.9	35.0	39.1	37.5	46.0	44.7	52.8	54.8	56.6	59.3
135	9.7	10.8	10.9	12.3	12.9	17.6	24.1	27.6	40.0	27.4	44.8	38.7	40.8	43.6	45.1
136	9.2	9.8	10.9	12.6	14.1	20.6	21.9	23.4	28.3	28.3	31.1	28.8	29.4	28.3	30.9
137	6.3	8.1	7.2	8.1	9.2	13.3	16.9	18.1	31.1	23.8	30.6	21.9	22.3	23.3	22.1
138	2.7	3.1	3.9	5.0	6.4	9.9	13.4	14.8	14.5	17.7	11.9	19.3	20.1	22.3	20.8
139	7.8	7.4	9.0	10.6	13.6	18.8	24.2	25.0	26.0	28.9	28.6	30.4	30.9	31.6	31.6
140	6.8	8.4	8.3	9.6	12.3	17.0	21.2	21.1	21.7	21.5	21.5	21.1	20.4	20.7	19.7
141	5.3	3.3	7.0	8.1	10.3	15.5	19.8	21.2	24.5	24.3	26.7	26.0	26.2	26.0	26.0

FIGURE 7.1

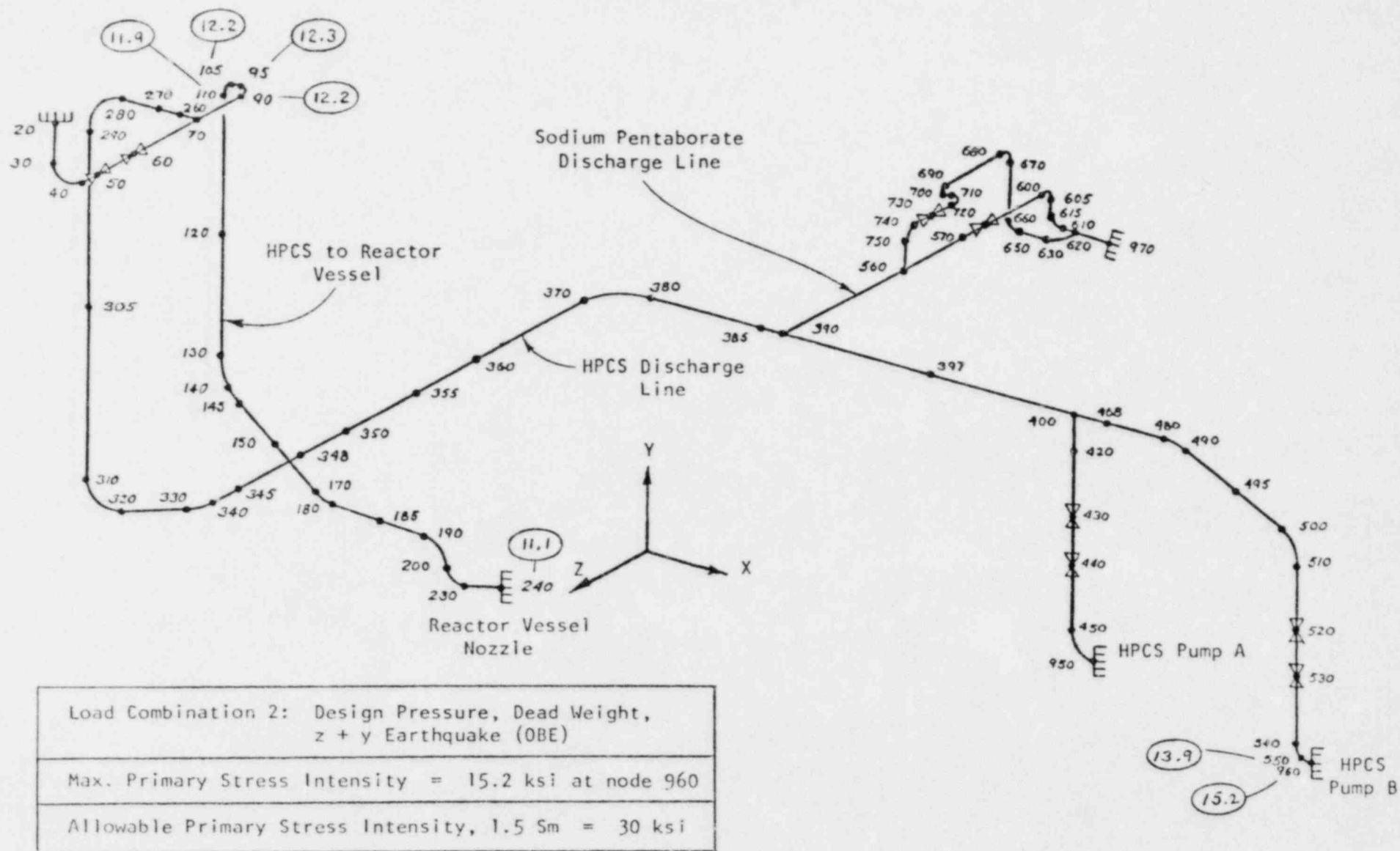
COMPLIANCE WITH ASME CODE EQUATION 9/DESIGN CONDITIONS



Load Combination I: Design Pressure, Dead Weight X + y Earthquake (OBE)
Max. Primary Stress Intensity = 12.7 ksi at node 960
Allowable Primary Stress Intensity, 1.5 S _m = 30 ksi

FIGURE 7.2

COMPLIANCE WITH ASME CODE EQUATION 9/DESIGN CONDITIONS



Load Combination 2: Design Pressure, Dead Weight, z + y Earthquake (OBE)
Max. Primary Stress Intensity = 15.2 ksi at node 960
Allowable Primary Stress Intensity, 1.5 S _m = 30 ksi

FIGURE 7.4

COMPLIANCE WITH ASME CODE EQUATION 10
NORMAL OPERATING CONDITIONS

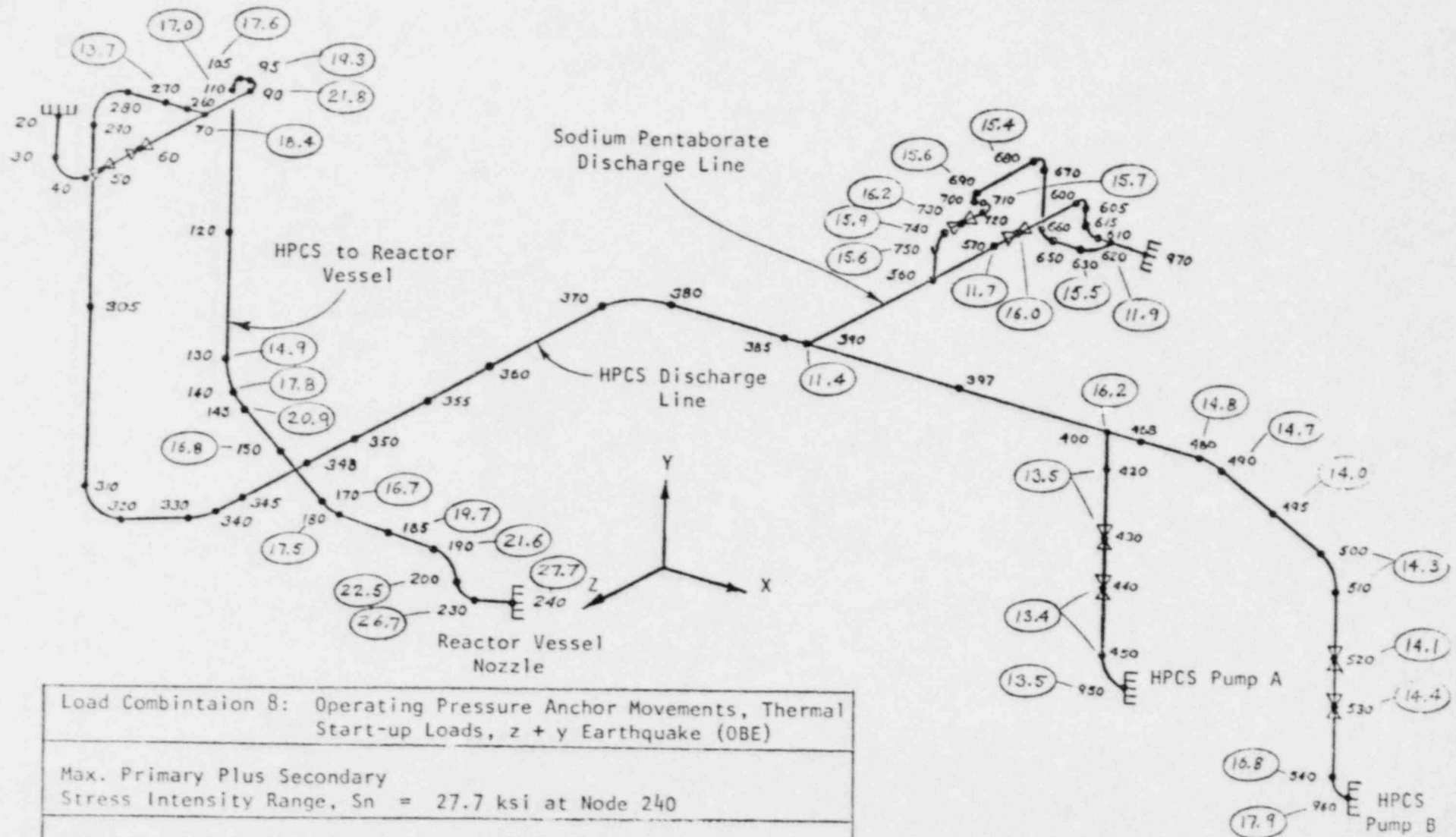
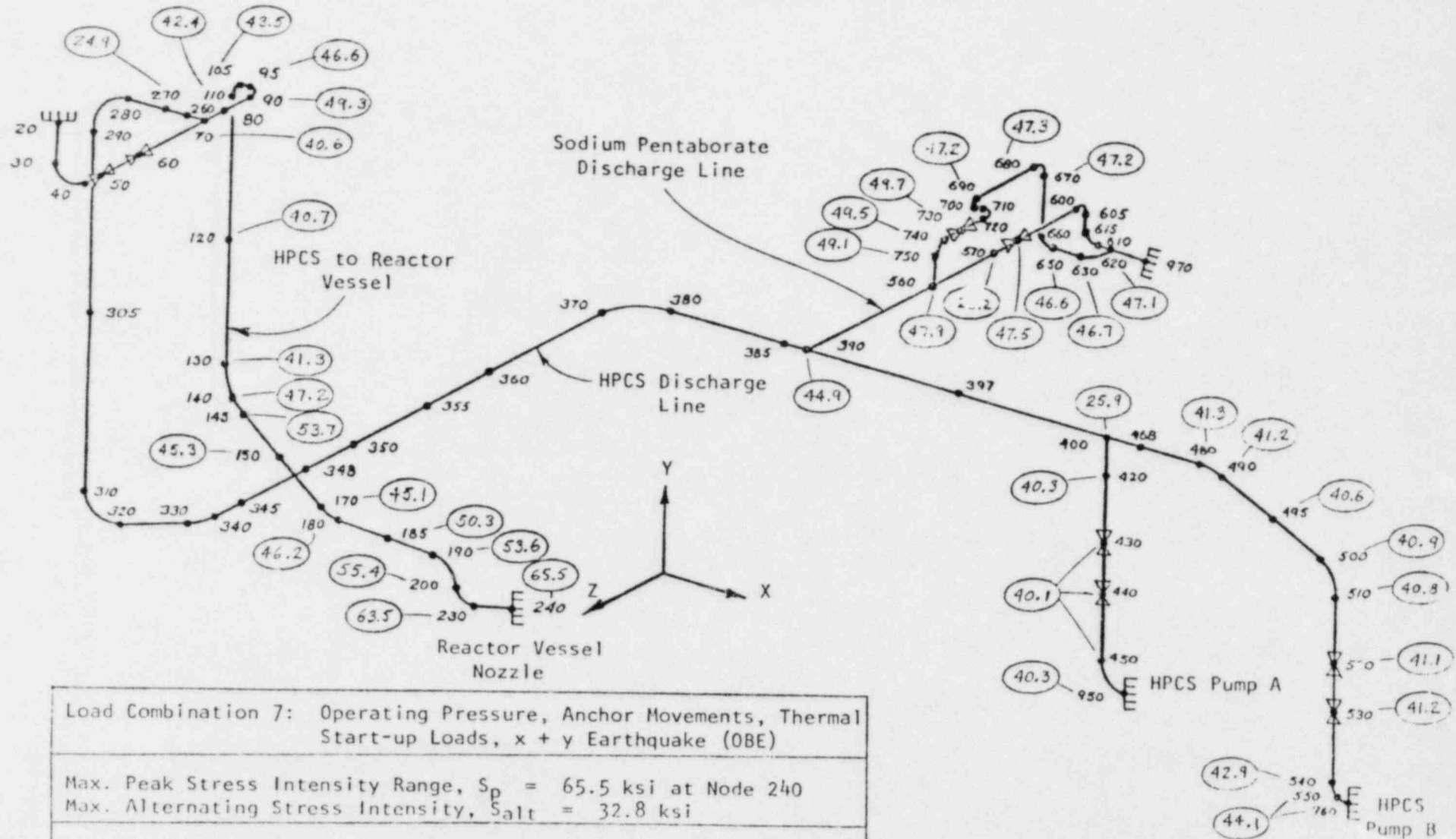


FIGURE 7.5

CONSIDERATION OF ASME CODE EQUATIONS 11 AND 14
NORMAL OPERATING CONDITIONS



Load Combination 7: Operating Pressure, Anchor Movements, Thermal Start-up Loads, x + y Earthquake (OBE)
Max. Peak Stress Intensity Range, $S_p = 65.5$ ksi at Node 240 Max. Alternating Stress Intensity, $S_{alt} = 32.8$ ksi
Max. Allowable Number of Stress Cycles, $N = 2 \times 10^5$

FIGURE 7.6

CONSIDERATION OF ASME CODE EQUATIONS 11 AND 14
NORMAL OPERATING CONDITIONS

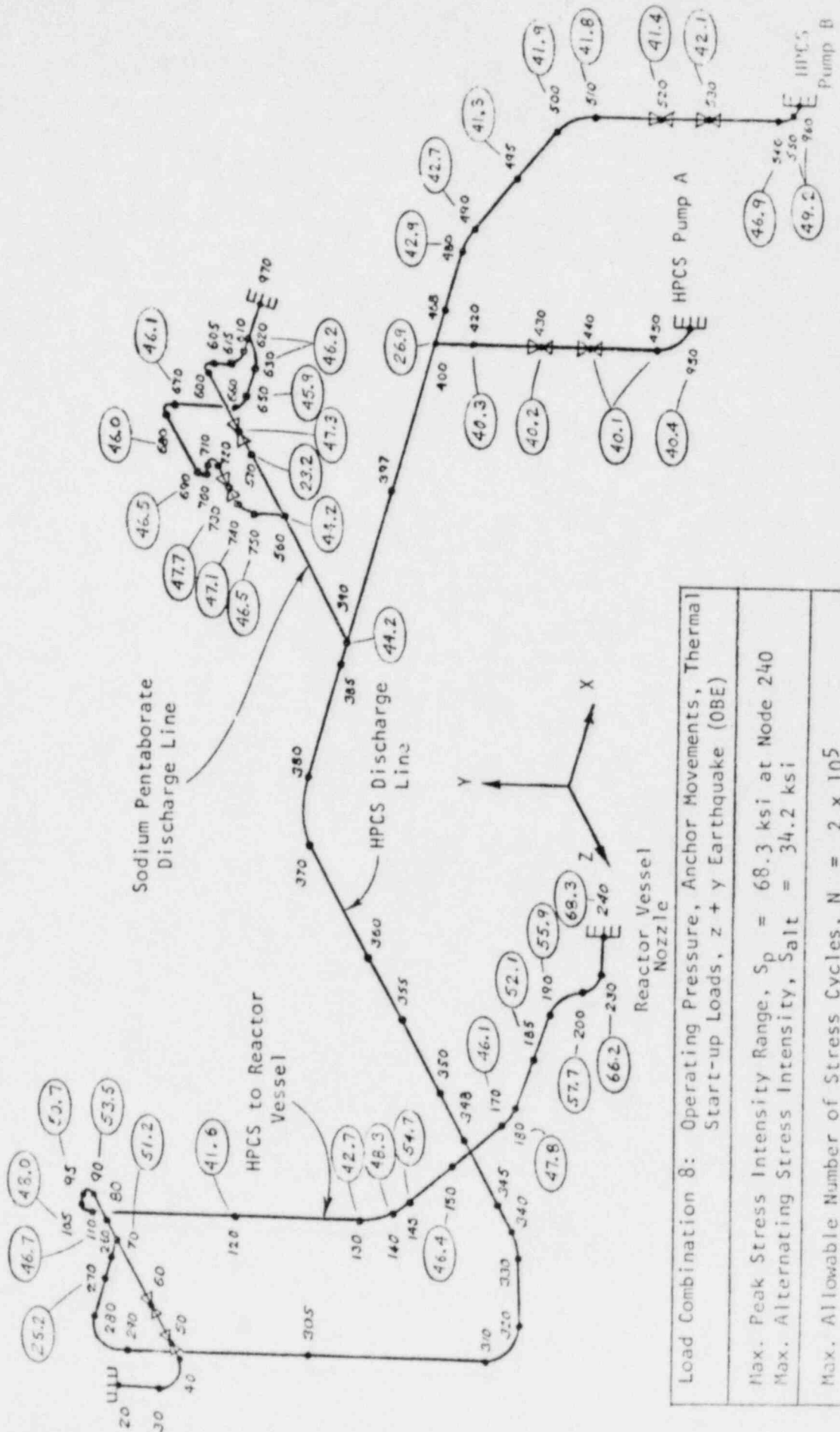
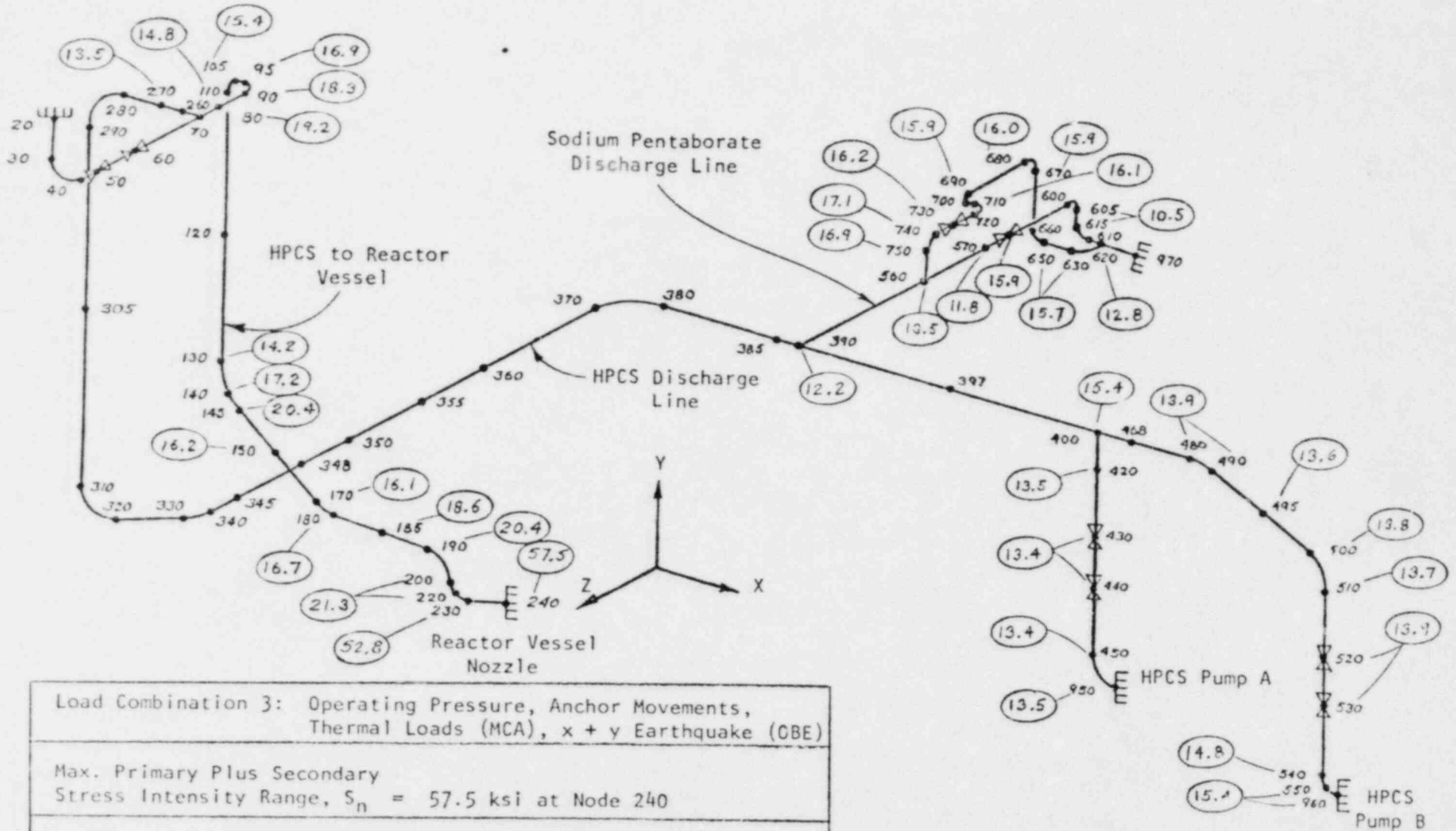


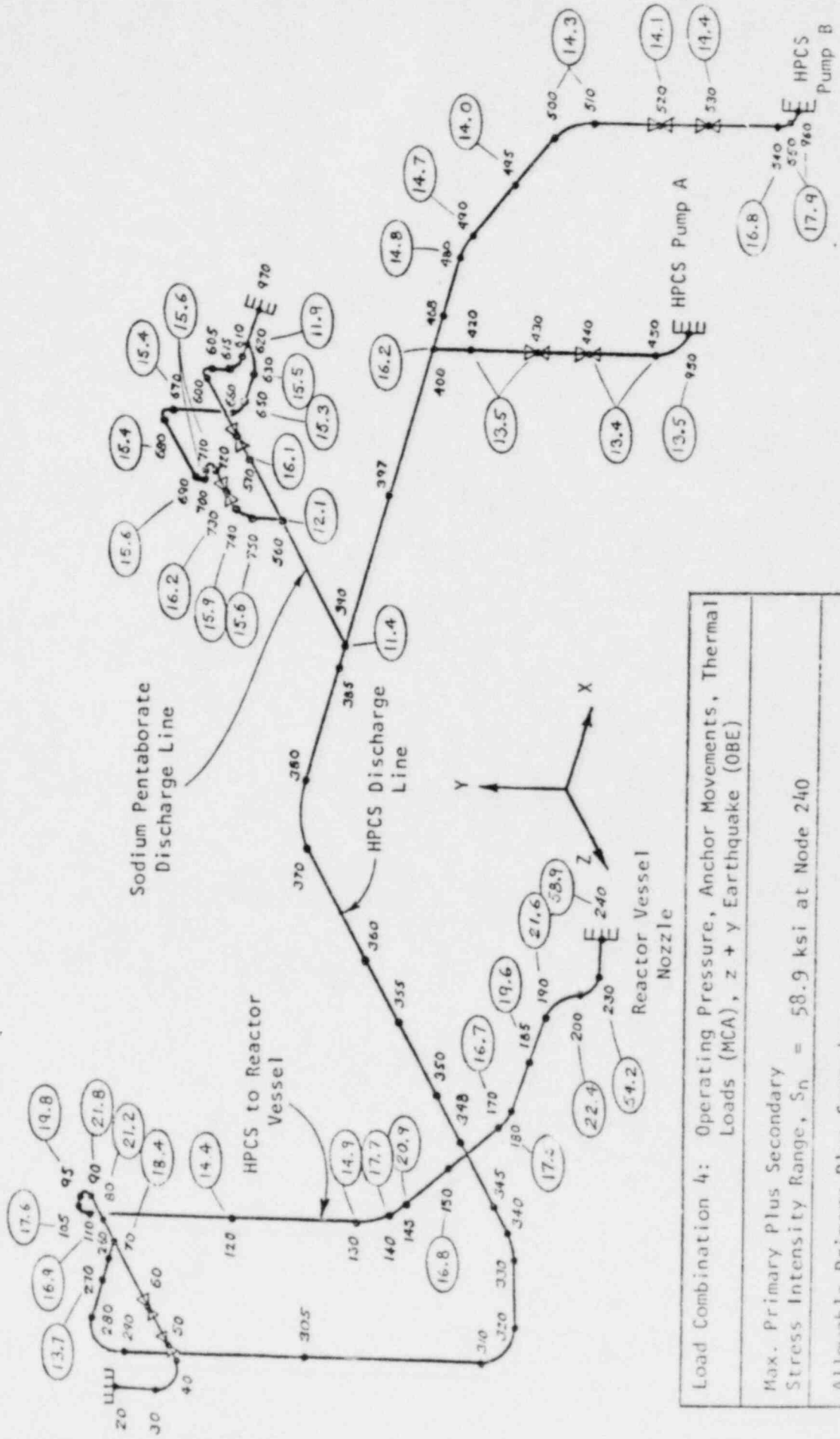
FIGURE 7.7

COMPLIANCE WITH ASME CODE EQUATION 10
UPSET CONDITIONS



Load Combination 3: Operating Pressure, Anchor Movements, Thermal Loads (MCA), x + y Earthquake (CBE)
Max. Primary Plus Secondary Stress Intensity Range, $S_n = 57.5$ ksi at Node 240
Allowable Primary Plus Secondary Stress Intensity Range, $3S_m = 60.0$ ksi

FIGURE 7.8
 COMPLIANCE WITH ASME CODE EQUATION 10
 UPSET CONDITIONS



Load Combination 4: Operating Pressure, Anchor Movements, Thermal Loads (MCA), z + y Earthquake (OBE)
Max. Primary Plus Secondary Stress Intensity Range, $S_n = 58.9$ ksi at Node 240
Allowable Primary Plus Secondary Stress Intensity Range, $3S_m = 60$ ksi

FIGURE 7.9

CONSIDERATION OF ASME CODE EQUATIONS 11 AND 14 UPSET CONDITIONS

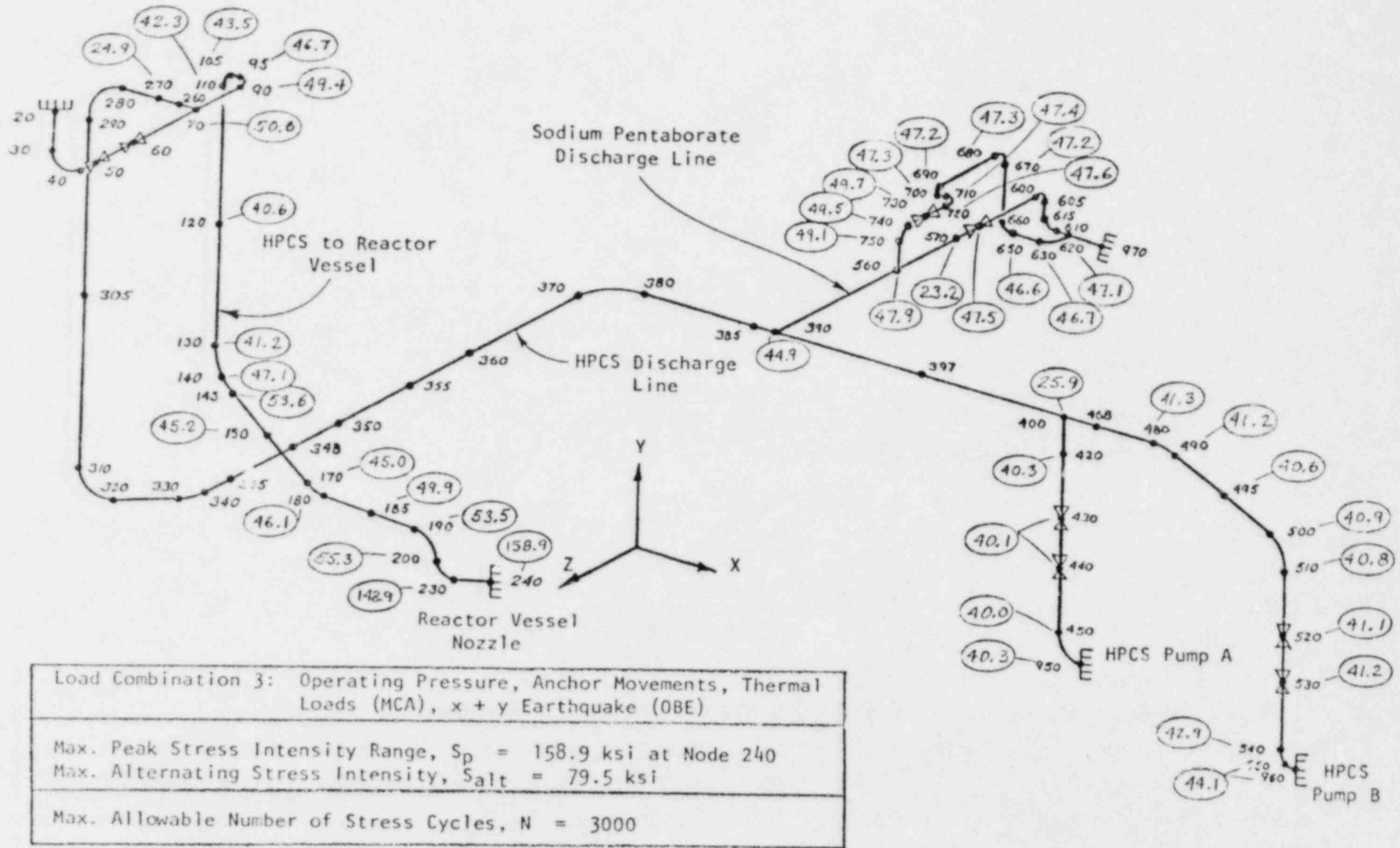


FIGURE 7.10

CONSIDERATION OF ASME CODE EQUATIONS 11 AND 14
UPSET CONDITIONS

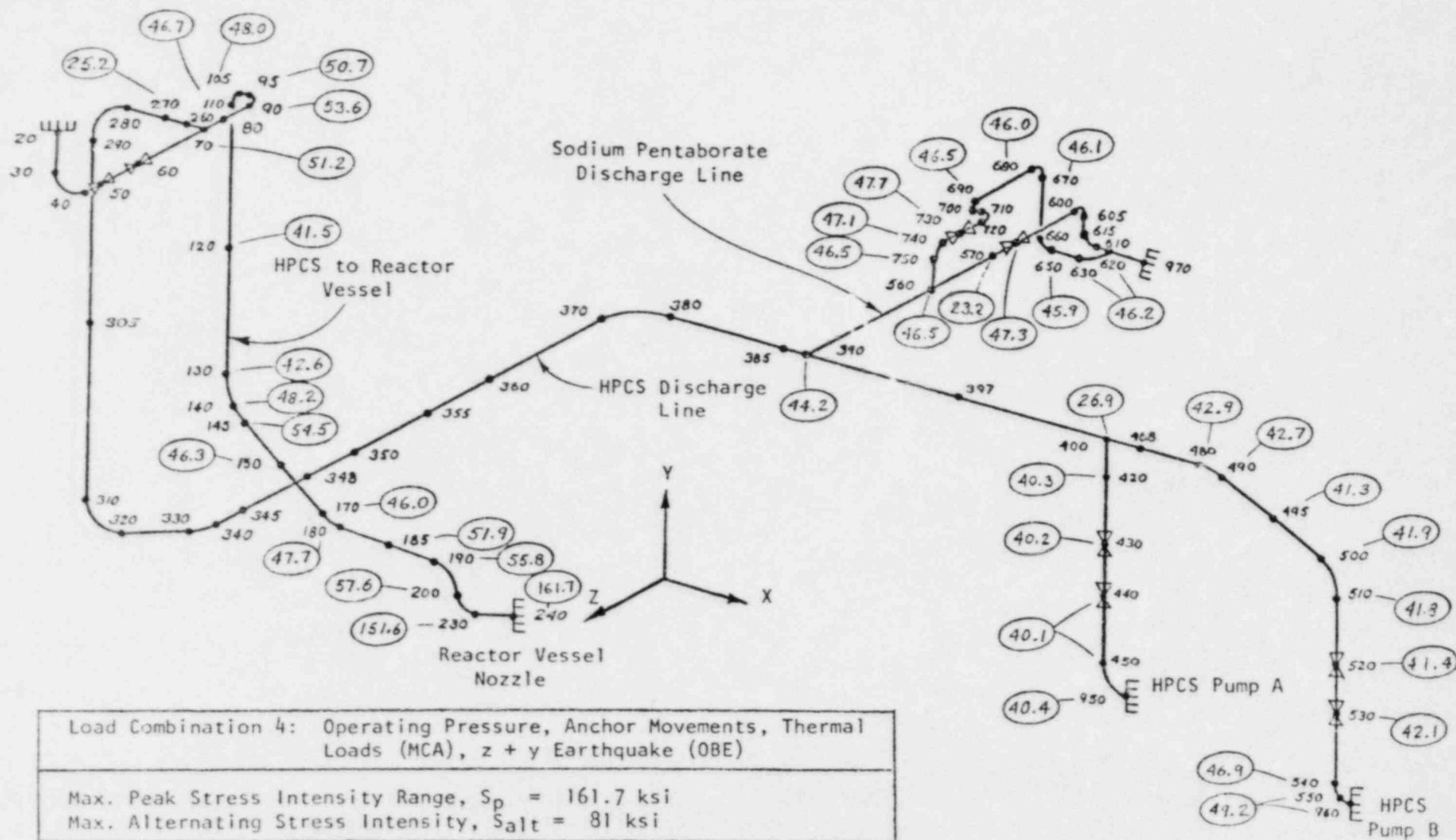


FIGURE 7.11

COMPLIANCE WITH ASME CODE EQUATION 9/FAULTED CONDITIONS

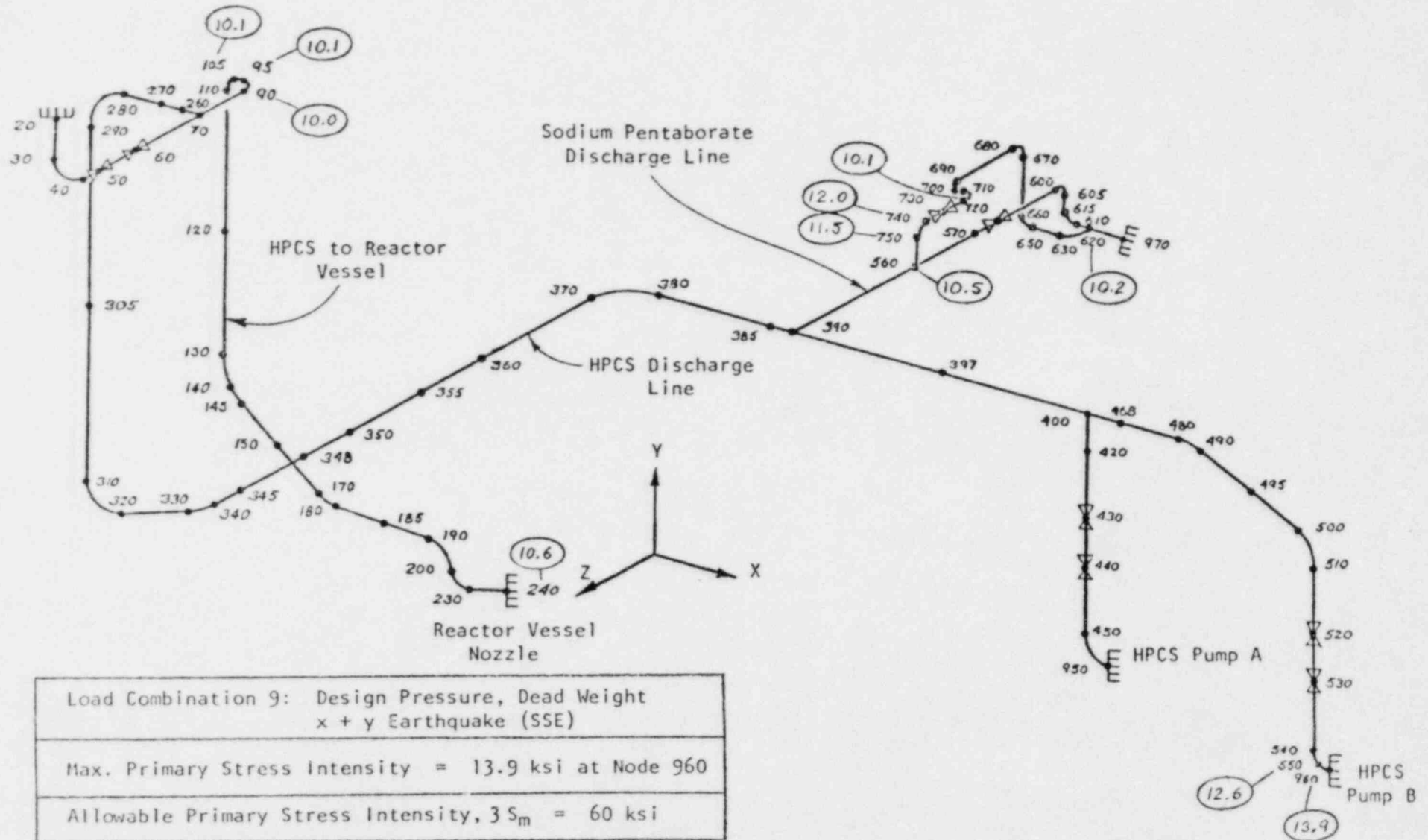
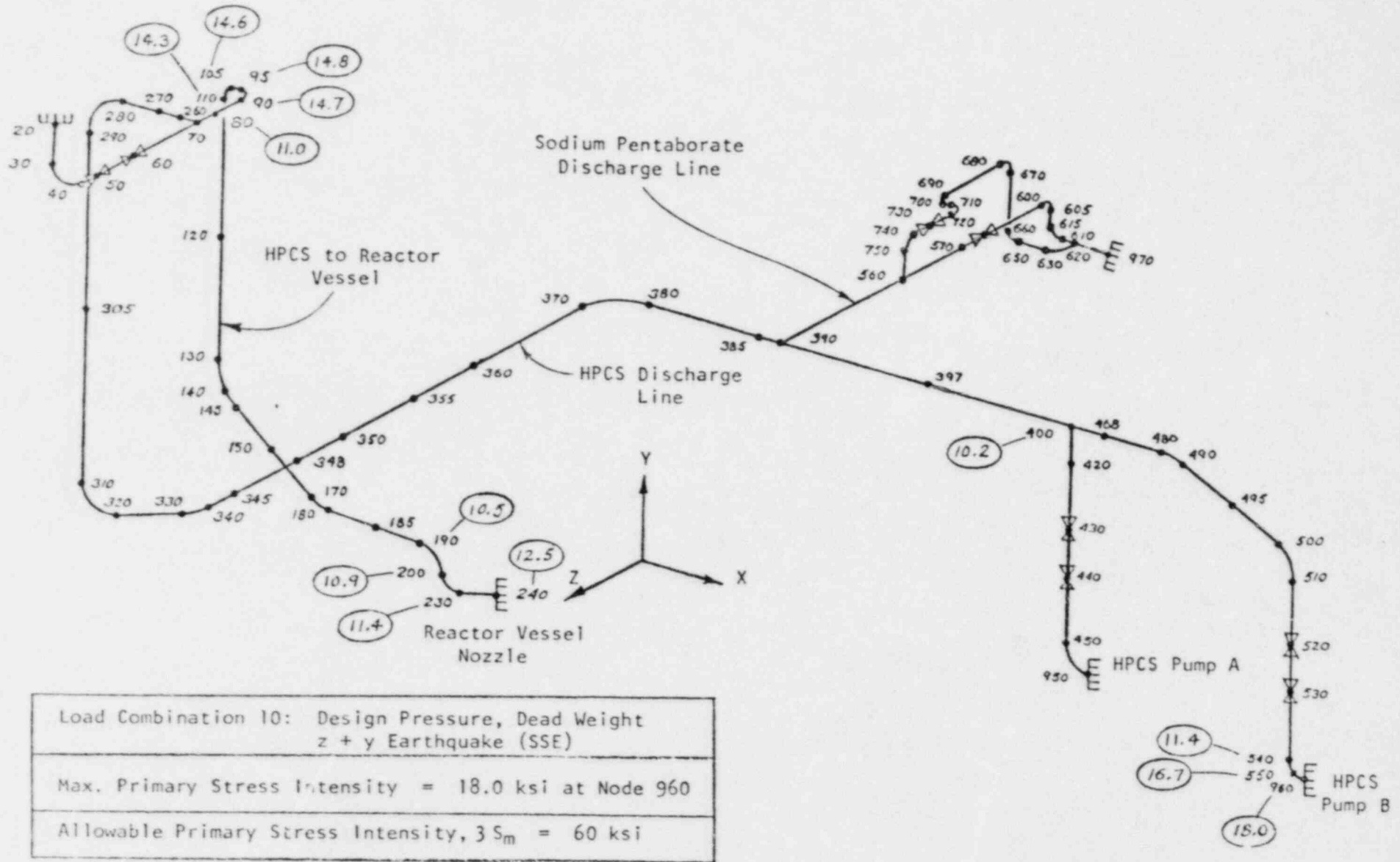


FIGURE 7.12

COMPLIANCE WITH ASME CODE EQUATION 9/FAULTED CONDITIONS



8. CONCLUSIONS

The results of the analysis indicate that the deflection of the HPCS discharge piping system due to dead weight, thermal loads and the specified seismic events are acceptable and that the stresses resulting from these loads, as calculated and combined in accordance with the rules given in Subarticle NB-3650 of Section III of the ASME Code (Reference 2), satisfy the design requirements for Class 1 piping systems provided that:

1. rigid anchors are provided as indicated by node points 20 and 970 of Figure 3.1,
2. the rotation of the eccentric actuator of the control valve CSV 204 (node point 50, Figure 3.1) is restrained by means of appropriate bars or struts,
3. the restraints are designed using the support reaction forces given in Table B-II of Appendix B, and
4. the total number of HPCS initiations is limited to 2900 cycles.*

NOTE: The HPCS system has operated (or cycled) a total of 226 times during all phases of plant testing and operation. However, only 25 operations or cycles have occurred with the LACBWR plant at or near operating temperature during its 7 year operating history. Considering a 40 year plant life, the total number of HPCS system operations with the plant at temperature is expected to be less than 150 cycles. Clearly this number is well below the maximum allowable number of cycles at operating temperature (2900 cycles).

9. REFERENCES

1. Gulf United Services Report No. SS-1162 "Seismic Evaluation of the LaCrosse Boiling Water Reactor", dated January 11, 1974.
2. ASME Boiler and Pressure Vessel Code, Section III, Division 1, 1974 Edition, Nuclear Power Plant Components.
3. Sargent and Lundy Engineers "Specification for Piping System-LaCrosse Boiling Water Reactor" LACBWR #256.
4. Sargent and Lundy Engineers "LACBWR" Project Drawing Nos. 41-503362 through 503378.
5. Allis-Chalmers, "LaCrosse Boiling Water Reactor Safeguards Report Volume I and II; LACBWR #283, dated August, 1967.
6. U.S. Atomic Energy Commission - Regulatory Guide 1.48, May, 1973.

APPENDIX A

LACBWR HPCS DISCHARGE LINE PIPING ANALYSIS
PIPESD ANALYTICAL INPUT DATA

<u>TABLE</u>		<u>PAGE</u>
A-I	Pipe Properties	A-1
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A-V	Dynamic Load Cases	A-6

TABLE A-1

PIPE DATA HPCS DISCHARGE LINE

RUN NO.	FROM POINT	TO POINT	PIPE SIZE	O.D. (in)	WALL THICKNESS (in)	MATERIAL (ASTM)	WEIGHT OF PIPE INCL. WATER (lb/ft)	WEIGHT OF INSULATION (lb/ft)	DESIGN TEMPERATURE (°F)	DESIGN PRESSURE (psig)
1	20	50	2-1/2" Sch 40	2.875	0.203	A312/TP304	7.87	1.01	120	100
2	230	240	1-1/2" Sch 80	1.900	0.200	A376/TP304	4.40	2.04	595	1400
3	50	80	2-1/2" Sch 80	2.875	0.276	A376/TP304	9.50	2.55	595	1400
3	80	230	1-1/2" Sch 80	1.900	0.200	A376/TP304	4.40	2.04	595	1400
4	70	250	2-1/2" Sch 80	2.875	0.276	A376/TP304	9.50	2.55	595	1400
4	250	260	2" Sch 80	2.375	0.218	A376/TP304	6.30	2.29	595	1400
4	260	270	2" Sch 80	2.375	0.218	A376/TP304	6.30	0.88	595	1400
4	270	470	2-1/2" Sch 80	2.875	0.276	A376/TP304	9.50	1.014	595	1400
5	390	570	2-1/2" Sch 80	2.875	0.276	A376/TP304	9.50	1.014	595	1400
5	570 730	580 560	2" Sch 80	2.375	0.218	A376/TP304	6.30	0.88	595	1400
5	580 620	590 730	2" Sch 80	2.375	0.218	A376/TP304	6.30	2.29	595	1400
5	590	970	2-1/2" Sch 80	2.875	0.276	A376/TP304	9.50	2.55	595	1400
6	400	950	1-1/2" Sch 80	1.900	0.200	A376/TP304	4.40	0.76	595	1400
7	470	960								

TABLE A-II
VALVE WEIGHTS

<u>VALVE</u>	<u>NODE LOCATION</u>	<u>TOTAL WEIGHT (lbs)</u>	<u>ECCENTRIC WEIGHT (lbs)</u>	<u>ECCENTRICITY (in)</u>
2-1/2" Control	50	285	145	21.72
2-1/2" Check	60	40	-	-
3-Way Control	260	215	125	21.72
1-1/2" Gate	430	52	-	-
1-1/2" Check	440	36	-	-
1-1/2" Gate	520	52	-	-
1-1/2" Check	530	36	-	-
2" Control	580	230	140	31.08
2" Control	730	230	140	31.08

TABLE A-111

STATIC LOAD CASES

PIPESD ANALYSIS

Static Load Case 1
Internal Design Pressure
Applied Pressure Loads:

<u>RUN NO.</u>	<u>NODES</u>	<u>PRESSURE (psig)</u>
1	20 to 50	100.0
2-7	All Others	1400.0

Static Load Case 2
Internal Operating Pressure
Applied Pressure Loads:

<u>RUN NO.</u>	<u>NODES</u>	<u>PRESSURE (psig)</u>
1	20 to 50	100.0
2,3	50 to 240	1340.0
4-7	All Others	1400.0

Static Load Case 3
Dead Weight
Applied Loads: 1.0G Vertical Acceleration

Static Load Case 4
Seismic Anchor Movements
Support Displacements:

<u>NODE</u>	<u>DIRECTION</u>	<u>DISPLACEMENT (in)</u>
240	X	0.432
240	Y	0.432

Static Load Case 5
 Thermal Anchor Movements
 Support Displacements:

<u>NODE</u>	<u>DIRECTION</u>	<u>DISPLACEMENT (in)</u>
240	X	-0.139
240	Y	1.260
240	Z	0.139

Static Load Case 8
 Thermal-Maximum Credible Accident
 Applied Thermal Loads:

<u>RUN NO.</u>	<u>NODES</u>	<u>TEMPERATURE CHANGE (°F)</u>
2	230-240	30.0

Discontinuity Stresses:

<u>NODE NO.</u>	<u>LINEAR TEMP. GRADIENT, T₁ (°F)</u>	<u>NON-LINEAR TEMP. GRADIENT, T₂ (°F)</u>	<u>THERMAL DISCONTINUITY STRESS (ksi)</u>
230	157.8	0.0	0.0
240	*	*	*

Static Load Case 9
 Thermal - Normal Start-Up And Shutdown
 Applied Thermal Loads:

<u>RUN NO.</u>	<u>NODES</u>	<u>TEMPERATURE CHANGE (°F)</u>
2	230-240	344.0

* Thermal Gradient and discontinuity stresses for node 240 are calculated in detailed ANSYS thermal stress analysis using the LION generated transient temperature profile (see Appendix B).

TABLE A-IV

SEISMIC RESPONSE SPECTRA PIPESD ANALYSIS

SPECTRUM NO. 1 (HORIZONTAL OBE)		SPECTRUM NO. 3 (HORIZONTAL SSE)	
FREQUENCY cps	ACCELERATION in/sec ²	FREQUENCY cps	ACCELERATION in/sec ²
40.00	61.82	40.00	108.19
8.00	61.82	10.00	103.19
5.98	104.33	6.57	127.51
5.20	185.47	5.40	251.84
4.20	471.41	4.69	258.89
3.80	564.14	4.20	492.66
3.35	197.06	4.09	444.36
2.95	100.46	4.00	467.54
2.39	197.06	3.59	376.74
2.25	176.74	3.40	289.80
2.10	320.71	2.95	173.88
2.00	251.16	2.50	318.78
1.62	838.48	2.29	289.80
1.60	637.56	2.10	502.32
1.58	683.93	2.00	405.72
1.37	239.57	1.75	1120.56
1.30	266.62	1.70	919.63
1.00	135.21	1.60	985.32
.83	83.07	1.35	434.70
.63	41.54	1.31	434.70
.50	1.54		

SPECTRUM NO. 2 (VERTICAL OBE)		SPECTRUM NO. 4 (VERTICAL SSE)	
FREQUENCY cps	ACCELERATION in/sec ²	FREQUENCY cps	ACCELERATION in/sec ²
40.00	15.84	40.00	31.68
33.00	15.84	33.33	32.68
20.00	28.98	20.00	51.00
14.92	40.96	14.92	68.01
10.00	67.23	9.00	108.19
9.00	77.28	2.39	132.14
2.25	92.73	2.00	106.64
1.65	66.07	1.49	81.14
1.00	39.80		
.80	35.16		
.59	25.50		
.40	17.00		
.25	10.82		
.20	7.23		
.10	1.16		

TABLE A-V
DYNAMIC LOAD CASES/PIPESD ANALYSIS

<u>Load Case No.</u>	<u>Load Description</u>	Spectrum No. In Global Direction		
		<u>X</u>	<u>Y</u>	<u>X</u>
6	x + y Earthquake (OBE)	1	2	0
7	z + y Earthquake (OBE)	0	2	1
10	x + y Earthquake (SSE)	3	4	0
11	z + y Earthquake (SSE)	0	4	3

APPENDIX B

LACBWR HPCS DISCHARGE LINE
PIPING ANALYSIS PIPESD ANALYSIS RESULTS

<u>TABLE</u>		<u>PAGE</u>
B-I	Joint Displacements	B-1
B-II	Support Reactions	B-8

TABLE B-1 (g)

JOINT DISPLACEMENTS (LOAD CASE 11)							JOINT /-----DISPLACEMENTS (IN.)-----/ /-----ROTATIONS (RADIANS)-----/						
SSS SEISMIC (Y+Z)							GID	X	Y	Z	X	Y	Z
TOTAL RESPONSE EQUALS MODE 1 THROUGH 27 BY SOSS SUMMATION							398	.0000010	.3005861	.0037273	.0009076	.0031474	.0071157
							400	.0000008	.0005507	.0026361	.0002129	.0003698	
							402	.0006975	.3005404	.0019553	.0001672	.0013445	.0001074
							410	.0013944	.0004608	.0022305	.0001458	.0001882	.0001074
							420	.0015571	.0002716	.0005109	.0001008	.0001816	.0001074
							430	.0000000	.3000000	.0000000	.0000000	.0000000	.0000000
							440	.0001915	.0002137	.0001714	.0000002	.0001428	.0001074
							445	.0001910	.0002021	.0000540	.0000182	.0001404	.0001074
							450	.0001744	.0002020	.0000725	.0000149	.0001354	.0001074
							460	.0000001	.3001122	.0000029	.0000017	.0001014	.0001074
							465	.0000004	.3000633	.0013911	.0000322	.0004392	.0001074
							468	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							470	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							480	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							490	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							500	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							510	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							520	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							530	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							535	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							540	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							550	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							555	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							559	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							560	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							562	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							570	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							575	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							580	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							581	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							590	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							600	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							605	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							615	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							617	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							619	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							620	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							625	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							625	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							630	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							640	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							645	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							590	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							660	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							670	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							680	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							685	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							690	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							700	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							710	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							720	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							730	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							731	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							740	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							745	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							750	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							950	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							960	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000
							970	.0000000	.0000000	.0000000	.0000000	.0000000	.0000000

TABLE B-11 (a)

E L A S T I C S U P P O R T R E A C T I O N S (LOAD CASE 3)

DEAD LOAD

SUPPORT JOINT	FORCE (LB.)			MOMENT (IN-LB.)		
	X	Y	Z	X	Y	Z
20	25.036	364.924	109.710	-121.400	-104.646	415.896
50	0.000	0.000	0.000	3484.668	-695.160	-9.000
145	0.000	46.398	0.000	0.000	0.000	0.000
240	-1.547	44.620	-0.43	-585.910	47.757	-868.011
260	-4.710	460.050	-115.688	0.000	0.000	0.000
345	-17.199	97.161	10.457	0.000	0.000	0.000
348	0.000	-27.372	0.000	0.000	0.000	0.000
355	0.000	17.152	0.000	0.000	0.000	0.000
385	0.000	151.419	0.000	0.000	0.000	0.000
430	17.665	0.000	4.880	0.000	0.000	0.000
468	-12.913	82.861	-8.434	0.000	0.000	0.000
495	0.000	28.418	0.000	0.000	0.000	0.000
575	0.000	-4.198	0.000	0.000	0.000	0.000
580	-5.959	328.359	52.060	0.000	0.000	0.000
730	19.379	353.534	-57.336	0.000	0.000	0.000
950	-12.123	158.335	-7.746	-7.633	-4.895	-1010.147
960	2.291	187.545	2.582	-1582.393	56.815	-88.369
370	-5.920	64.279	2.558	-81.755	-139.362	-1125.066

TABLE B-11 (b)

E L A S T I C S U P P O R T R E A C T I O N S (LOAD CASE 4)

SEISMIC ANCHOR MOVEMENTS - X AND Z (NODE 240) OBE

SUPPORT JOINT	/-----FORCE (LB.)-----/			/-----MOMENT (IN-LB.)-----/		
	X	Y	Z	X	Y	Z
20	6.5257	-33.5644	-30.8358	339.1796	-27.2763	108.4049
50	0.0000	0.0000	0.0000	-883.3631	-181.2601	-182.8113
145	0.0000	-12.2452	0.0000	0.0000	0.0000	0.0000
240	20.9915	-11.0890	13.6953	272.7224	730.1319	172.1798
260	-26.3573	54.4503	17.6553	0.0000	0.0000	0.0000
345	-1.4461	5.2041	-.1792	0.0000	0.0000	0.0000
348	0.0000	-2.9037	0.0000	0.0000	0.0000	0.0000
355	0.0000	.1845	0.0000	0.0000	0.0000	0.0000
385	0.0000	-.0238	0.0000	0.0000	0.0000	0.0000
430	.0189	0.0000	.0030	0.0000	0.0000	0.0000
468	.2229	.0042	.0206	0.0000	0.0000	0.0000
495	0.0000	.0066	0.0000	0.0000	0.0000	0.0000
575	0.0000	.0149	0.0000	0.0000	0.0000	0.0000
580	.0033	-.0130	-.3334	0.0000	0.0000	0.0000
730	.0577	.0100	-.0296	0.0000	0.0000	0.0000
950	-.0016	-.0367	-.0005	-.0057	.0378	.2877
960	.0000	-.0018	-.0014	-.0491	-.0092	-.0002
970	-.0150	.0031	.0058	.0173	.0388	.1159

TABLE B-11 (c)

E L A S T I C S U P P O R T R E A C T I O N S

(LOAD CASE 5)

THERMAL ANCHOR MOVEMENTS

SUPPORT JOINT	FORCE (LB.)			MOMENT (IN-LB.)		
	X	Y	Z	X	Y	Z
20	-9.409	-36.374	1.292	-191.655	39.330	-156.309
50	0.000	0.000	0.000	-717.347	261.423	544.365
145	0.000	-142.590	0.000	0.000	0.000	0.000
240	-73.313	45.967	-11.378	111.000	-444.797	-2512.800
260	25.973	150.060	16.448	0.000	0.000	0.000
345	7.519	-29.263	-7.275	0.000	0.000	0.000
348	0.000	12.852	0.000	0.000	0.000	0.000
355	0.000	-.889	0.000	0.000	0.000	0.000
385	0.000	.131	0.000	0.000	0.000	0.000
430	-.111	0.000	-.017	0.000	0.000	0.000
468	-.506	-.020	-.049	0.000	0.000	0.000
495	0.000	-.057	0.000	0.000	0.000	0.000
575	0.000	-.045	0.000	0.000	0.000	0.000
580	.055	.004	.845	0.000	0.000	0.000
730	-.247	-.065	.151	0.000	0.000	0.000
950	.003	.260	.003	.030	-.054	-1.945
960	-.002	.019	.003	-.073	.022	.015
970	.038	.009	-.022	-.205	.161	.371

TABLE B-II (d)

E L A S T I C S U P P O R T R E A C T I O N S

(LOAD CASE 6)

03E SEISMIC - X+Y

TOTAL RESPONSE EQUALS MODE 1 THROUGH 27 BY SQSS SUMMATION

SUPPORT JOINT	/-----FORCE (LB.)-----/			/-----MOMENT (IN-LB.)-----/		
	X	Y	Z	X	Y	Z
20	16.925	11.550	7.789	153.8	69.5	279.2
50	0.000	0.000	0.000	293.1	191.8	256.0
145	0.000	7.479	0.000	0.0	0.0	0.0
240	7.203	4.467	7.193	111.7	399.7	181.2
260	19.334	15.678	20.989	0.0	0.0	0.0
345	13.291	11.560	9.322	0.0	0.0	0.0
348	0.000	7.971	0.000	0.0	0.0	0.0
355	0.000	9.278	0.000	0.0	0.0	0.0
385	0.000	9.132	0.000	0.0	0.0	0.0
430	2.456	0.000	1.577	0.0	0.0	0.0
468	23.622	4.519	10.921	0.0	0.0	0.0
495	0.000	3.889	0.000	0.0	0.0	0.0
575	0.000	25.678	0.000	0.0	0.0	0.0
580	21.531	42.449	11.552	0.0	0.0	0.0
730	91.047	9.573	18.014	0.0	0.0	0.0
950	.854	7.747	.291	2.9	13.1	53.9
960	11.788	3.010	13.495	493.8	173.7	246.5
970	5.677	7.463	6.790	90.7	186.2	651.5

TABLE B-11 (e)

ELASTIC SUPPORT REACTIONS (LOAD CASE 7)

OBE SEISMIC - Y+Z

TOTAL RESPONSE EQUALS MODE 1 THROUGH 27 BY SOSS SUMMATION

SUPPORT JOINT	FORCE (LB.)			MOMENT (IN-LB.)		
	X	Y	Z	X	Y	Z
20	14.260	16.582	9.515	198.	59.	236.
50	0.000	0.000	0.000	373.	238.	284.
145	0.000	8.165	0.000	0.	0.	0.
240	7.895	4.890	13.792	141.	929.	262.
260	15.511	20.652	23.697	0.	0.	0.
345	8.651	12.701	5.732	0.	0.	0.
348	0.000	7.333	0.000	0.	0.	0.
355	0.000	8.116	0.000	0.	0.	0.
385	0.000	6.555	0.000	0.	0.	0.
470	2.423	0.000	3.319	0.	0.	0.
468	6.717	9.971	18.399	0.	0.	0.
495	0.000	2.577	0.000	0.	0.	0.
575	0.000	63.262	0.000	0.	0.	0.
580	5.321	46.160	41.625	0.	0.	0.
730	7.256	8.525	58.847	0.	0.	0.
950	1.461	10.087	.600	6.	21.	60.
960	14.462	4.761	30.913	1138.	326.	486.
970	2.628	16.368	3.560	43.	46.	264.

TABLE B-II (g)

E L A S T I C S U P P O R T R E A C T I O N S

(LOAD CASE 11)

SSE SEISMIC (Y+7)

TOTAL RESPONSE EQUALS MODE 1 THROUGH 27 BY SOSS SUMMATION

SUPPORT JOINT	FORCE (LB.)			MOMENT (IN-LB.)		
	X	Y	Z	X	Y	Z
20	24.14	25.29	16.49	331.	99.	399.
50	0.00	0.00	0.00	578.	376.	455.
145	0.00	12.58	0.00	0.	0.	0.
240	12.91	7.55	20.70	218.	1375.	396.
260	25.05	31.88	38.43	0.	0.	0.
345	15.25	19.64	9.49	0.	0.	0.
348	0.00	11.52	0.00	0.	0.	0.
355	0.00	13.07	0.00	0.	0.	0.
385	0.00	10.89	0.00	0.	0.	0.
430	3.94	0.00	5.38	0.	0.	0.
468	11.44	16.16	29.79	0.	0.	0.
495	0.00	4.38	0.00	0.	0.	0.
575	0.00	110.60	0.00	0.	0.	0.
580	9.03	80.65	72.55	0.	0.	0.
730	11.05	14.89	102.82	0.	0.	0.
950	2.38	16.35	.97	10.	33.	97.
960	23.50	7.82	50.09	1840.	527.	786.
970	4.58	28.63	6.21	75.	80.	460.

APPENDIX C

LACBWR HPCS DISCHARGE NOZZLE WELD ANSYS ANALYSIS INPUT DATA

TABLE C-1 Element Properties

FIGURE C-1 Thermal Transient Loads
(From Appendix E Results)

TABLE C-1
ANSYS MODEL ELEMENT PROPERTIES
HPCS DISCHARGE SOCKET WELD

ELEMENT NUMBERS	MATERIAL	PROPERTY	TEMPERATURE, °F					
			70	100	200	300	400	500
1-31 33-41 43-48 50-55 57-59 64-77 86-88 93-95 100-102 105-110	STAINLESS STEEL SA376, SA312 TP304H	Elastic Modulus $E, 10^6$ Psi	← 27.0 →					
		Design Stress Intensity S_m, ksi	20.0	20.0	20.0	20.0	18.7	17.4
		Mean Coeff. Of Thermal Expansion, $\alpha, 10^{-6} \text{ } ^\circ\text{F}^{-1}$	9.11	9.16	9.34	9.47	9.59	9.70
		Poisson's Ratio ν	← 0.30 →					
82-84 89-91 96-98 103 111-146	INCONEL 600 SB 166	$E, 10^6$ Psi	← 30.0 →					
		S_m, ksi	← 23.3 →					
		$\alpha, 10^{-6}, \text{ } ^\circ\text{F}^{-1}$	7.13	7.20	7.40	7.56	7.70	7.8
		ν	← 0.30 →					
42, 49, 56 60-63 78-81 85, 92, 99, 104	DUMMY GAP	$E, 10^{-6}$ Psi	← 0.10 →					
		$\alpha, 10^{-6} \text{ } ^\circ\text{F}^{-1}$	← 0.10 →					
		ν	← 0.30 →					

FIGURE C-1 (cont'd)
ANSYS THERMAL TRANSIENT LOADING

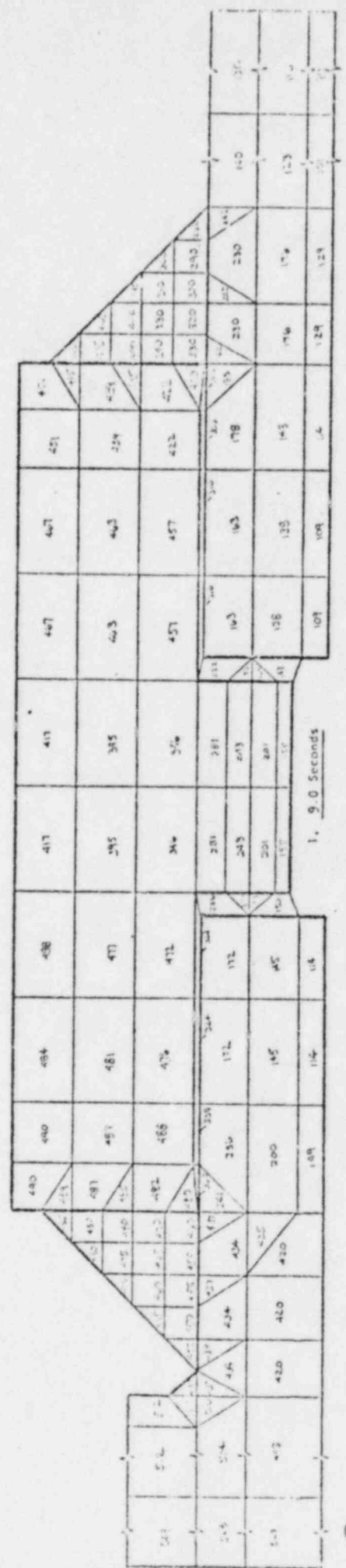
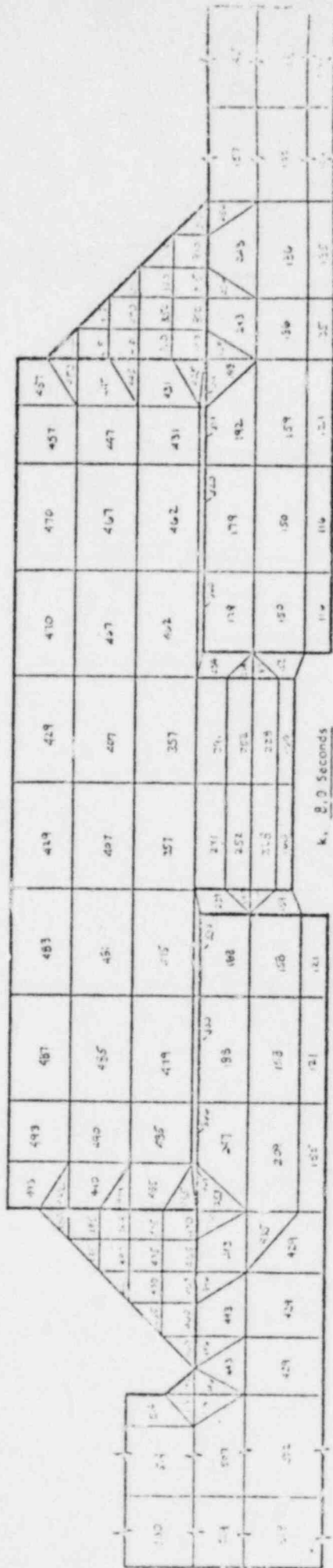


FIGURE (cont'd)
ANSYS THERMAL TRANSIENT LOADING

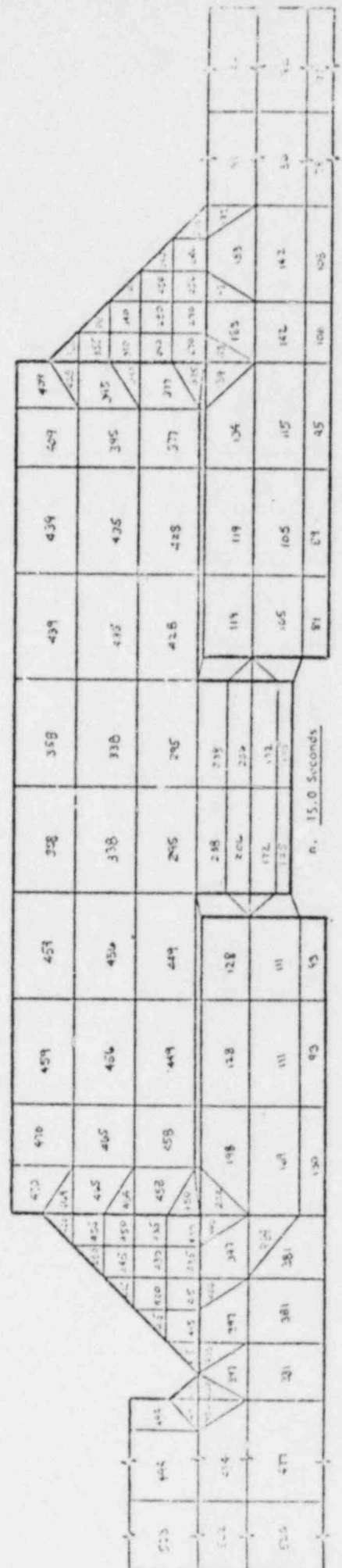
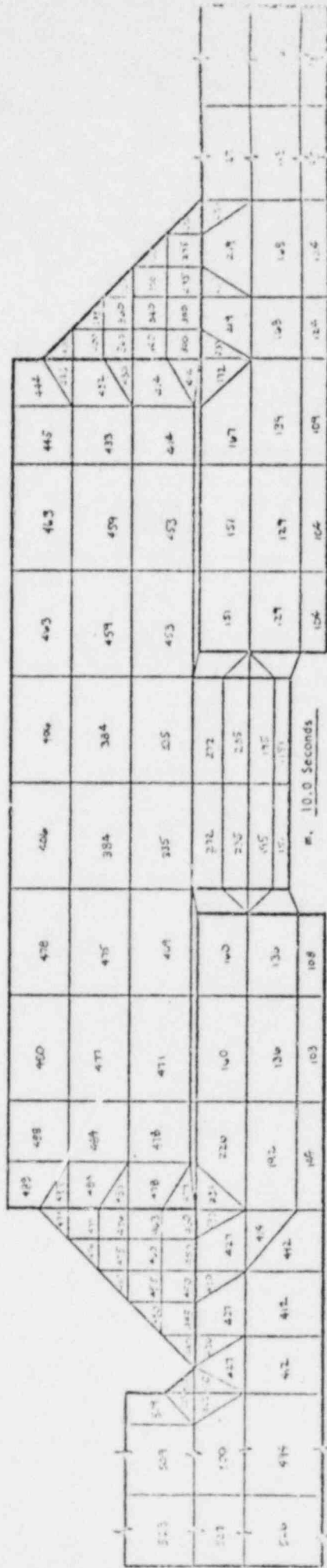
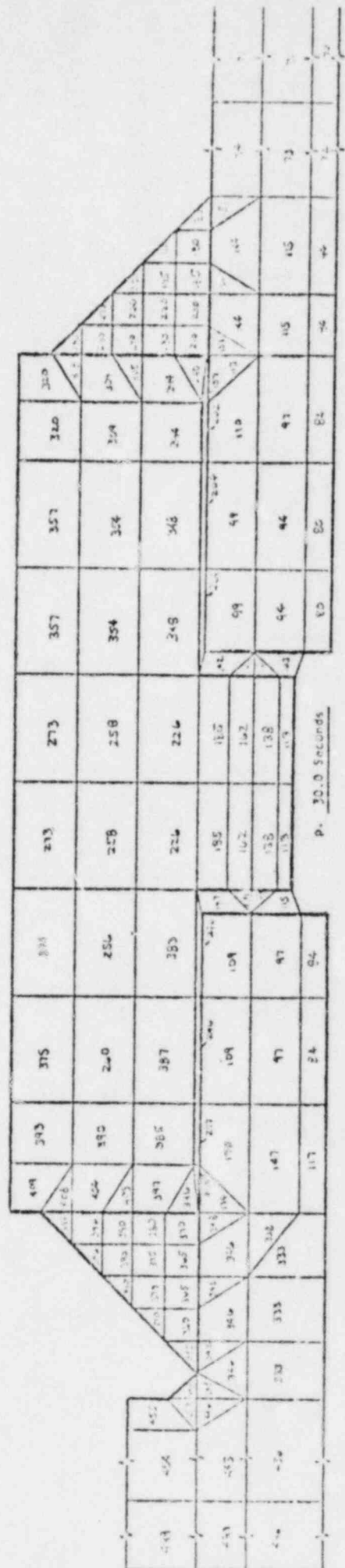
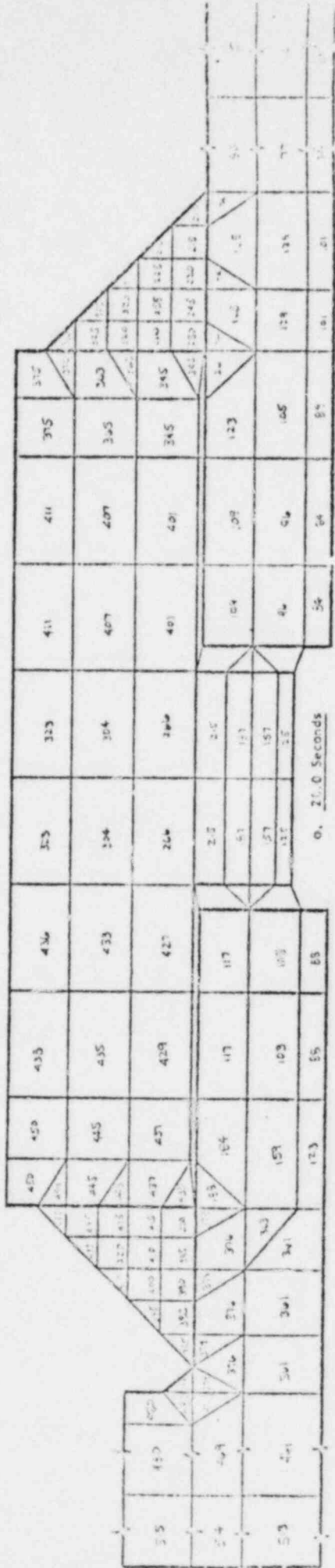


FIGURE 4 (cont'd)
ANSYS THERMAL TRANSIENT LOADING



APPENDIX D

ASME CODE CALCULATIONS AT HPCS DISCHARGE NOZZLE

(Refer to Code Calculations for Node 240, PIPESD Piping
Analysis NES Binder S-03)

Consideration of Design Conditions

The primary stress intensity limit is satisfied if the requirement of Equation 9 is met.

$$B_1 \frac{PD_o}{2t} + B_2 \frac{D_o}{2I} M_i \leq 1.5S_m \quad (9)$$

where:

B_1, B_2 = primary stress indices for the specific product under investigation.

P = design pressure, psi

D_o = outside diameter of pipe, in.

t = nominal wall thickness of product, in.

S_m = allowable design stress intensity value, psi

I = moment of inertia, in⁴

M_i = resultant moment loading due to loads caused by (1) weight, (2) earthquake, considering only one-half the range of the earthquake and excluding the effects of anchor displacement due to earthquake, and (3) other sustained design mechanical loads.

From computer output S-03

For x + y Earthquake Equation 9 is satisfied as

$$4.99 + 4.87 \leq 1.5 \times 20 \quad (9)$$

$$\text{or } 9.5 \leq 30.0 \text{ ksi}$$

For y + z Earthquake Equation 9 is satisfied as

$$4.99 + 6.08 \leq 1.5 \times 20 \quad (9)$$

$$11.07 \leq 30.0 \text{ ksi}$$

Consideration of Normal Conditions

Satisfaction of Primary Plus Secondary Stress Intensity Range

Primary plus secondary stress intensity range is calculated using Equation (10)

$$S_n = C_1 \frac{P_o D_o}{2t} + C_2 \frac{D_o}{2l} M_i + \quad (10)$$
$$\frac{1}{2(1-\nu)} E\alpha |\Delta T_1| + C_3 E_{ab} |\alpha_a T_a - \alpha_b T_b| \leq 3S_m$$

(Refer to ASME Code for nomanclature)

Since thermal stress components of equation (10) are not evaluated in the detail transient ANSYS stress analyses, they will be conservatively estimated from the equation (11) stress analysis results.

From Computer Output S-03

$$K_1 C_1 \frac{P_o D_o}{2t} = 38.19 \text{ ksi}$$
$$K_2 C_2 \frac{D_o}{2l} M_i = 27.24 \text{ ksi for } x + y \text{ Earthquake}$$
$$= 29.99 \text{ ksi for } z + y \text{ Earthquake}$$

From detail transient ANSYS stress analysis for thermal loadings: the maximum peak stress intensity due to structural and material discontinuity and linear and non-linear temperature gradient is:

$$\frac{1}{2(1-\nu)} K_3 E\alpha |\Delta T_1| + K_3 C_3 E_{ab} |\alpha_a T_{aa} - \alpha_b T_b| +$$
$$\frac{1}{1-\nu} E\alpha |\Delta T_2| = 93.5 \text{ ksi in ANSYS}$$

Model Element No. 39

For Socket Weld fitting, stress indices K_1 , K_2 and K_3 are

$$K_1 = 3.0; K_2 = 2.0; K_3 = 3.0$$

$$C_1 \frac{P_o D_o}{2t} = \frac{38.19}{3} = 12.73 \text{ ksi}$$

$$C_2 \frac{D_o}{2l} = \frac{27.24}{2} = 13.62 \text{ ksi for } x + y \text{ Earthquake}$$
$$= \frac{29.99}{2} = 15.0 \text{ ksi for } z + y \text{ Earthquake}$$

And Conservatively:

$$\frac{1}{2(1-\nu)} E\alpha |\Delta T_1| + C_3 E_{ab} |\alpha_a T_a - \alpha_b T_b|$$

$$= \frac{93.5}{3} = 31.17 \text{ ksi}$$

∴ Equation (10) at Node 240 is satisfied as

$$12.73 + 13.62 + 31.17 \leq 3 \times 20.0$$

$$57.52 \leq 60.0 \quad (10) \quad \text{For } x + y \text{ Earthquake}$$

$$12.73 + 15.0 + 31.17 \leq 3 \times 20.0$$

$$58.90 \leq 60.0 \text{ ksi} \quad (10) \quad \text{For } z + y \text{ Earthquake}$$

Satisfaction of Peak Stress Intensity Range

Peak Stress Intensity Range S_p is calculated using Equation (11)

$$S_p = K_1 C_1 \frac{P_0 D_0}{2t} + K_2 C_2 \frac{D_0}{2l} M_i +$$

$$\frac{1}{2(1-\nu)} K_3 E\alpha |\Delta T_1| + K_3 C_3 E_{ab} |\alpha_a T_a - \alpha_b T_b| + \frac{1}{1-\nu} E\alpha |\Delta T_2| \quad (11)$$

From Computer Output S - 03

$$K_1 C_1 \frac{P_0 D_0}{2t} = 38.19 \text{ ksi}$$

$$K_2 C_2 \frac{D_0}{2l} M_i = 27.24 \text{ ksi for } x + y \text{ Earthquake}$$

$$= 29.99 \text{ ksi for } z + y \text{ Earthquake}$$

From detail transient and ANSYS stress analysis for thermal loading

$$\frac{1}{2(1-\nu)} K_3 E\alpha |\Delta T_1| + K_3 C_3 E_{ab} |\alpha_a T_a - \alpha_b T_b| + \frac{1}{1-\nu} E\alpha |\Delta T_2|$$

$$= 93.5 \text{ ksi in ANSYS Model}$$

Element No. 89

$$\therefore S_p = 38.19 + 27.24 + 93.5 = 158.93 \text{ ksi for } x + y \text{ Earthquake}$$

$$S_p = 38.19 + 29.99 + 93.5 = 161.68 \text{ ksi for } z + y \text{ Earthquake}$$

Simplified Elastic - Plastic Discontinuity Analysis

Since Equation (10) is satisfied, Equations (12) and (13) need not be evaluated.

Alternating Stress Intensity

The alternating stress intensity S_{alt} is calculated using Equation (14)

$$S_{alt} = K_e \frac{S_p}{2} \quad (14)$$

where:

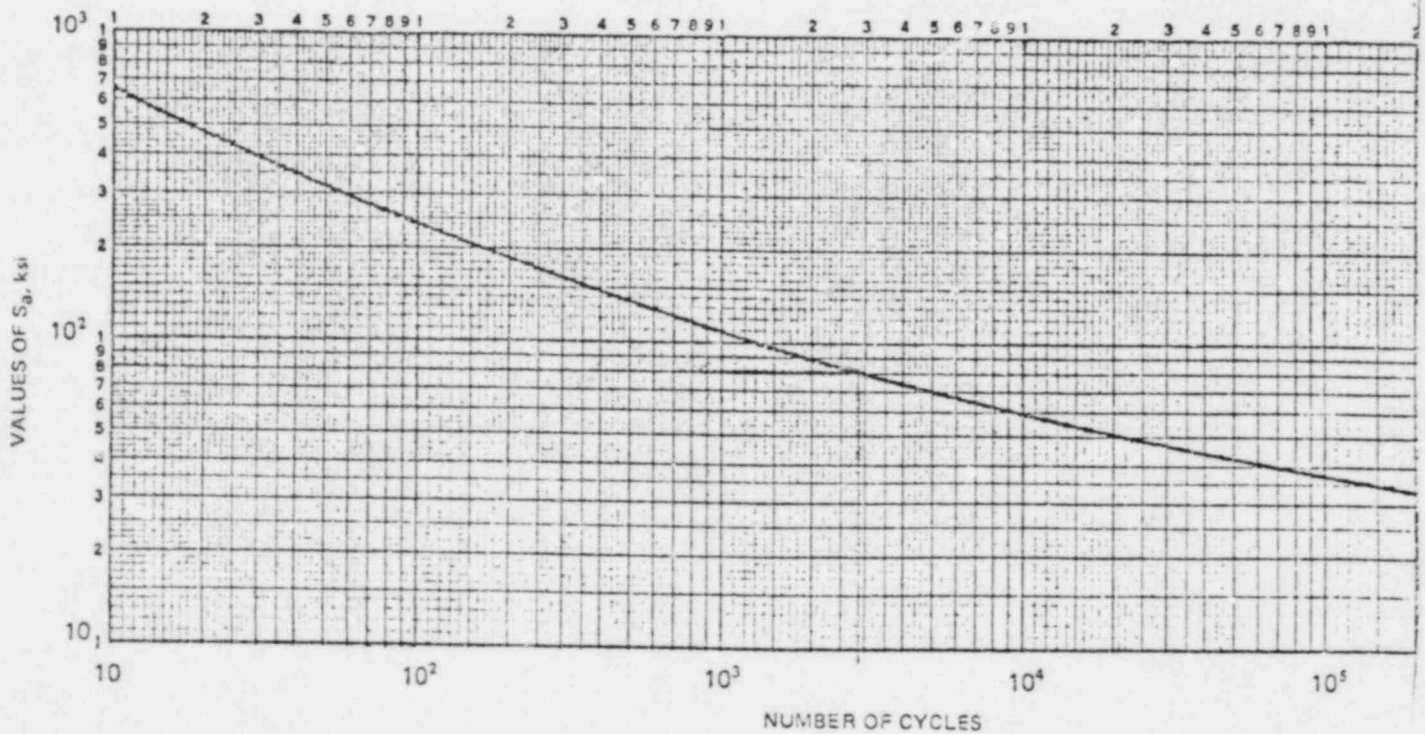
S_{alt} = alternating stress intensity

S_p = peak stress intensity value calculated by Equation (11)

$K_e = 1.0$ for $S_n \leq 3 S_m$

$$\begin{aligned} \therefore S_{alt} &= 1.0 \times \frac{158.93}{2} = 79.47 \text{ ksi for } x + y \text{ Earthquake} \\ &= 80.84 \text{ ksi for } z + y \text{ Earthquake} \end{aligned}$$

\therefore From Figure I-9.2 of the ASME Code the allowable number of cycles is 2900.



NOTE:
E = 26.0 x 10⁶ psi

FIG. I-9.2 DESIGN FATIGUE CURVE FOR AUSTENITIC STEELS, NICKEL-CHROMIUM-IRON ALLOY, NICKEL-IRON-CHROMIUM ALLOY AND NICKEL-COPPER ALLOY

APPENDIX E

HPCS DISCHARGE NOZZLE THERMAL
TRANSIENT ANALYSIS WITH THE LION 4 CODE



LACROSSE HPCS SEISMIC THERMAL STUDY

1. PURPOSE - To COMPUTE Steady STATE AND TRANSIENT TEMPERATURES IN LACROSSE Boiling Water Reactor Vessel Nozzle AND HPCS PIPE, DURING EMERGENCY CORE COOLING.

2. SUMMARY - The LION 4 (ref 1) THERMAL-HYDRAULICS CODE WAS USED TO COMPUTE THE TEMPERATURES FOR LACROSSE IN THE HPCS PIPE DURING EMERGENCY COOLING.

A THERMAL MODEL WAS DEVELOPED TO FIND THE STEADY STATE TEMPERATURE DISTRIBUTION AND THE RESULTS WERE INPUT AS THE INITIAL TEMPERATURES IN THE TRANSIENT ANALYSIS FOR MAXIMUM CREDIBLE ACCIDENT (HPCS WATER FLOW AT 70°F, AND 577.5°F TO 270°F AT -20°F/SEC IN THE CORE; REF. 4). THE CRITICAL SECTION OF THE MODEL IS SHOWN IN FIGURES 1 AND 2.

A SERIES OF TEMPERATURE MEANS AND DIFFERENCES WERE OBTAINED TO BE USED IN THERMAL STRESS CALCULATIONS.



3. Analysis - A) Materials - The pipe, coupling, and welds were all assumed to have the properties of stainless steel. The nozzle is a ferritic steel forging with an inconel extension welded to the forging, and clad with stainless steel (Ref 2., and Ref 4. fig A-1B)
- B) Steady State - In constructing the steady state model, the coolant was treated as a solid mass, represented by internal nodes, and the steady state temperature distribution obtained was used as input for the transient analysis. The results of the steady state analysis are shown in Figure 3.
- C) Transients - In the transient analysis, the internal nodes, representing the coolant, were removed, and replaced by coolant nodes. A boundary node was placed at the entrance to the channel to ensure constant inlet temperature. Flow is assumed to start instantaneously at time = 0.0 min. The transient analysis corresponds to conditions for maximum credible accident as



specified in LACEWR Safeguard Report (Ref. 4), which specifies a 20° F/sec. vessel temperature change. The transient temperatures are shown in Figures 4 thru 30

D) TEMPERATURE RANGES AND DIFFERENCES:

T_A = MEAN TEMP. THRU THE THICKNESS OF MATERIAL

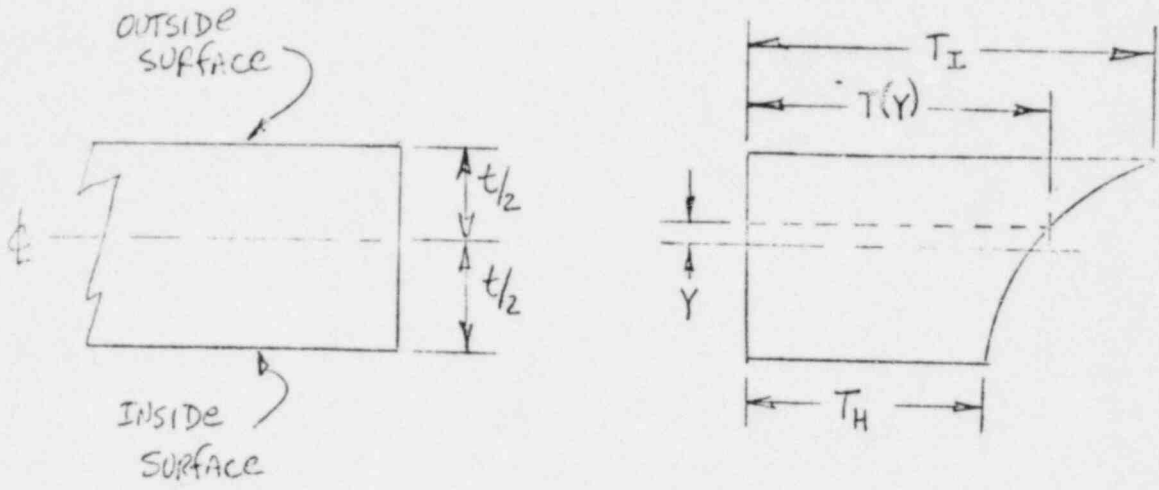
$|\Delta T_1|$ = RANGE OF ABSOLUTE VALUE OF THE TEMP. DIFFERENCE BETWEEN THE OUTSIDE SURFACE (T_o) AND THE INSIDE SURFACE (T_i) OF THE MATERIAL, ASSUMING MOMENT-GENERATING EQUIVALENT TEMPERATURE DISTRIBUTION (REF. 3, PG 31).

$|\Delta T_2|$ = RANGE OF ABSOLUTE VALUE FOR THAT PORTION OF THE NON-LINEAR THERMAL GRADIENT THROUGH THE WALL, NOT INCLUDED IN $|\Delta T_1|$ (REF. 3, PG 64).

FURTHER, ΔT_1 IS DEFINED BY:

$$\Delta T_1 = \left(\frac{12}{t^2} \right) \int_{-t/2}^{t/2} y T(y) dy$$

; see Fig A (below) and Ref. 3, pg 57.



T_I = OUTSIDE SURFACE TEMP.
 T_H = INSIDE SURFACE TEMP.
 $T(y)$ = TEMP. AT POINT (y)
 t = WALL THICKNESS

FIGURE A. WALL MODEL DEFINITIONS.



ΔT_2 is defined by:

$$\Delta T_2 = \text{MAX} \left[|T_A - T_I| - \left| \frac{\Delta T_1}{2} \right|, |T_H - T_A| - \left| \frac{\Delta T_1}{2} \right|, 0 \right]; (\text{Ref 3, pg 58})$$

where T_H & T_I ARE INSIDE AND OUTSIDE SURFACE TEMPERATURES, RESPECTIVELY, AND T_A IS:

$$T_A = \frac{1}{t} \int_{-t/2}^{t/2} T(y) dy$$

ΔT_1 AND ΔT_2 ARE FURTHER ILLUSTRATED IN FIG. B, BELOW.

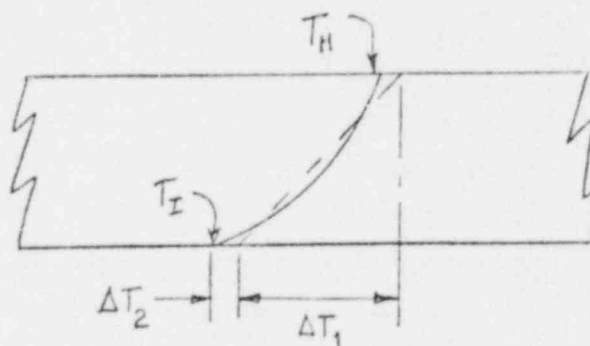


FIG. B. GRAPHIC PRESENTATION OF
LINEAR AND NON-LINEAR THERMAL
GRADIENT THRU WALL THICKNESS
(REF. 3, PG 36)



4. RESULTS: The steady state axial temperatures obtained from Ref. 5, are shown in Figure 3.

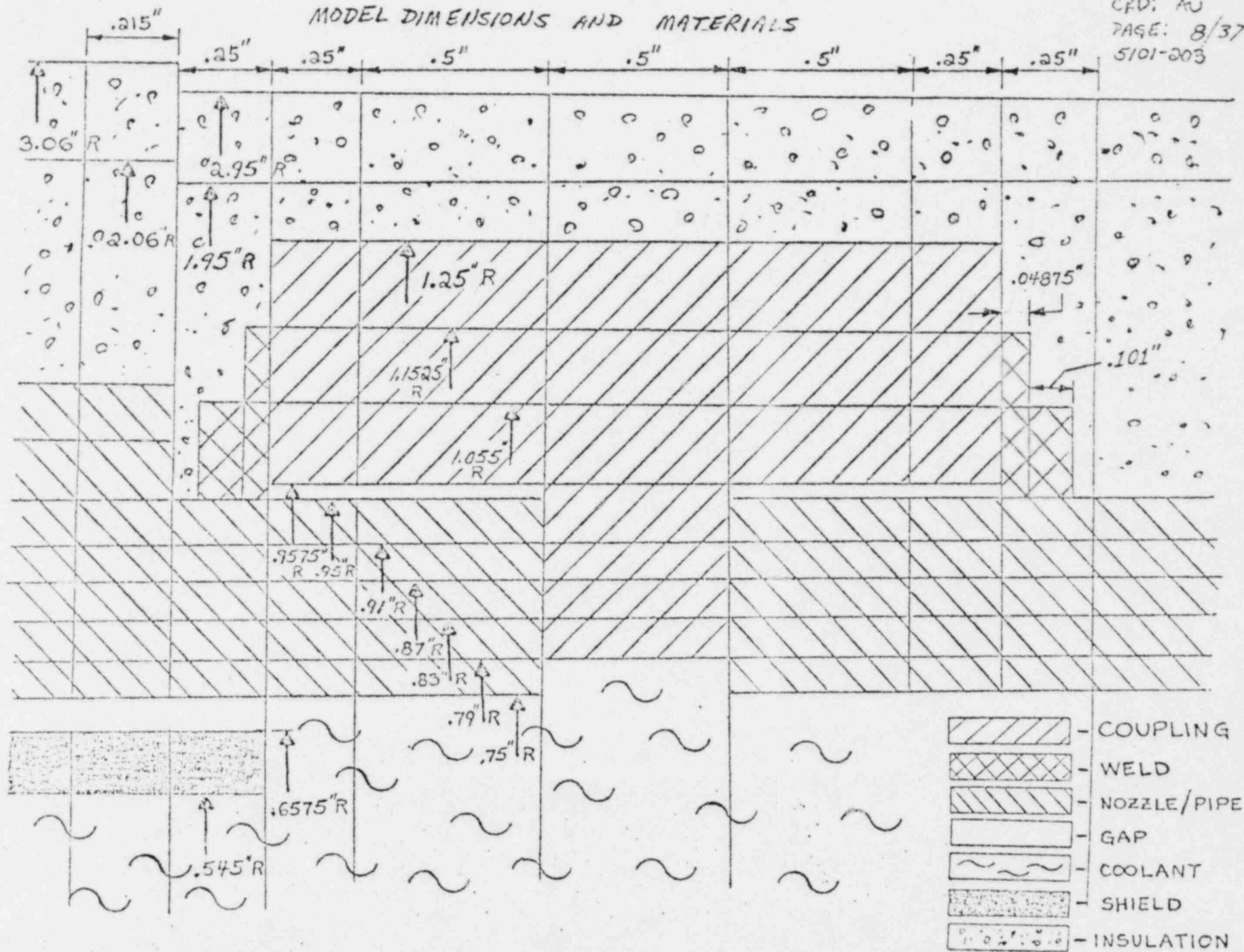
The transient temperature distributions obtained from Ref. 6, are shown in Figures 4 thru 30.

In addition, normal startup and shutdown transients with 150°F/hr rate of temperature changes have been considered in the thermal shock analysis. The steady state temperature distribution of Figure 3 represents the most conservative axial temperature profile during these transients and therefore were considered as such. ΔT_1 and ΔT_2 values are negligible during startup and shutdown transients (less than 2°F).

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FIGU. 752

MODEL DIMENSIONS AND MATERIALS



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FIGURE 3 - STEADY STATE TEMPERATURE DISTRIBUTION

529	523	507	501	494	488	481	476	473	472	451
529	523	505	501	494	488	481	476	473	472	451
529	522	509	505	498	488	479	474	471	471	451
529	522	510	505	498	488	479	474	470	470	451
529	521	510	505	498	488	479	474	470	470	451
529	521	511	505	498	488	479	474	470	470	451
529	521	511	505	498	488	479	474	470	470	451
540	539	539								

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FIGURE 21 - NODE TEMPERATURES AT TIME = 7.5 SEC

526	516	466	472	487	435	472	460	320	169
526	514	472	486	484	413	459	450	353	264
526	510	487	479	363	297	404	435	238	163
525	508	260	202	257	192	193	206	209	152
525	506	250	192	212	168	168	177	176	135
524	505	230	176	163	147	147	154	176	113
524	504	200	154	126	121	121	125	139	
524	504	159	126						
84	84	85							



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- 2) LETTER TO I. HUSSAIN FROM H.A. TOWSLEY, DAIRYLAND POWER COOPERATIVE, NO. IAC-3477, OCT. 29, 1975.
- 3) ORNL-TM-3645, "NUCLEAR PIPING DESIGN", ORNL RIDGE NATIONAL LABORATORY AND TELETYPE MATERIALS RESEARCH (UNDER SUBCONTRACT NO. 3059 WITH UNION CARBIDE CORP., NUCLEAR DIVISION), FEB. 1972.
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