1 2	UNITED STATES OF AMERICA NUCLEAR REGULATORY COMMISSION
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0	BEFORE THE ATOMIC SAFETY AND LICENSING BOARD APR 2319
3	97 Office of the & S
4	In the Matter of
5	HOUSTON LIGHTING & POWER COMPANY) Docket No. 50-466
6 7	(Allens Creek Nuclear Generating) Station, Unit No. 1)
8 9	DIRECT TESTIMONY OF KAMRAN MOKHTARIAN ON BEHALF OF HOUSTON LIGHTING & POWER COMPANY ON DOHERTY CONTENTION NO. 9-CONTAINMENT BUCKLING
10	Q. Please state your name and place of employment.
11	A. My name is Kamran Mokhtarian. I am employed by Chicago
12	Bridge & Iron Company. My business address is 800 Jorie
13	Boulevard, Oak Brook, Illinois.
14	Q. Please describe your professional qualifications.
15	A. A statement of my background and qualifications is
16	attached as Exhibit KM-1.
17	Q. Wiy have you prepared this testimony?
18	A. The purpose of this testimony is to address Doherty's
19	Contention No. 9 which alleges that the Applicant's steel
20	containment shell will not be strong enough to resist
21	buckling under the design loads. Doherty's Contention No. 9
22	alleges:
23	That Intervenor's health and safety interests are
24	inadequately protected because Applicant's steel containment shell is not strong enough by design
25	to resist dynamic and static loads which may plausibly occur in the life of the atomic plant.
26	The only specific basis stated in the contention for the
27	above allegations are four observations on containment
28	vessel bucking evaluation methods paraphrased from a

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2	preliminary (Jan. 1978) report of an NRC consultant, namely:
3	(1) Adequate experimental data for determining design
4	criteria did not exist.
5	(2) Computer programs for determining building loads
6	do not predict experimental buckling results very well.
7	(3) That the ASME Section III Buckling Criteria
8	Regulatory Guide 1.57 NE-3224 (sic) "permits designers
9	to select the method which yields a buckling stress
10	which is least conservative."
11	(4) Until more test data is obtained to study the
12	effects of imperfections, asymmetric loading, load
13	interaction, dynamic and nonlinear effects, a con-
14	servative factor of safety such as 3 should be used."
15	Q. Will you describe how the containment for Allens Creek
16	is being designed?
17	A. The steel containment vessel for ACNGS, as specified in
18	Subsection 3.8 of the PSAR, is being designed in accordance
19	with the requirements of the American Society of Mechanical
20	Engineers Boiler and Pressure Vessel Code (ASME Code)
21	Section III, Subsection NE. Chicago Bridge & Iron Company
22	(CBI) is designing the steel containment vessel and its
23	appurtenances for the ACNGS. The Applicant, through Ebasco,
24	has prepared the design specification required by Paragraph
25	NA-3250 of the ASME Code for use by CBI in their design of
26	the ACNGS steel containment vessel and its appurtenances.
27	This design specification establishes the minimum requirements
28	for the design of the vessel. These requirements include

2 the identification of the load definitions and the establish-3 ment of appropriate load combinations and related acceptance 4 criteria to be employed in assessing structural stability 5 and buckling capacity.

CBI is performing the required analyses and design 6 activities to configure the steel containment vessel which 7 will comply with the Applicant's design specification. CBI 8 upon completion of their ongoing design activities, will 9 prepare and submit to the Applicant a Certified Stress 10 Report in accordance with Article NA-3350 of the ASME Code. 11 How does this design process account for buckling? 12 Q. The PSAR Table 3.8-2 Itlines the buckling criteria in 13 A. use for ACNGS. This criteria is based on the classical 14 linear theory with reductions applied to account for imper-15 fections in vessel geometry and other differences between 16 theoretical and actual load capacities. 17

18 Basically, the method used on ACNGS for the buckling 19 evaluation is the following:

The containment vessel is mathematically modeled 20 1. using Kalnins' Shells of Revolution Program which has been 21 verified as producing results for axisymmetric shells 22 comparable to those of finite element programs recommended 23 in NUREG/CR-0793. The Kalnins' Program is based on linear 24 theory. The loads, as specified for ACNGS, are imposed on 25 this mathematical model of the containment vessel in accord-26 ance with the specified loading combinations. The program 27 has capabilities for axisymmetric and nonaxisymmetric stress 28

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analyses of axisymmetric shell structures.

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2. For the buckling analysis, the maximum compressive stresses at any azimuth are assumed to act uniformly all the way around, resulting in a conservative analysis.

The maximum stresses resulting from the sum of 3. 6 the static and dynamic loads will be compared to critical 7 buckling stresses using the specified stress interaction 8 equations which include the appropriate factors of safety. 9 This method of analysis accounts for the amplification 10 factors on stresses due to dynamic loadings. These resulting 11 stresses, however, are treated as equivalent static stresses 12 for comparison with critical buckling stresses. This is a 13 conservative approach, since a structure can withstand 14 stresses due to dynamic loadings that are equal to or, in 15 many cases, greater than critical stresses from statically 16 applied loadings. 17

18 The buckling capacity of the shell is based on linear 19 bifurcation (classical) analyses reduced by capacity reduction 20 factors which account for the effects of imperfections and 21 nonlinearity in geometry and boundary conditions and by 22 plasticity reduction factors which account for nonlinearity 23 in material properties.

In addition to the above reduction factors, factors of safety are employed in the assessment of structural stability. A factor of safety of 2.75 is applied wherever the critical buckling stresses are in the elastic range. The safety factor is linearly reduced from 2.75 to 2.0 between the

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proportional limit and the yield stress of the material. Where the critical stresses approach the yield strength of the material, material deformation becomes the controlling factor rather than buckling.

In addition to meeting the requirements of PSAR Table
3.8-2, the design of ACNGS containment vessel will meet the
requirements of ASME Code Case N-284, titled "Metal Containment Shell Buckling Methods," issued August 25, 1980.
Q. What do you understand to be the basis for Mr. Doherty's
contention?

Mr. Doherty filed, as a basis for his contention on 12 Α. containment buckling, his summary of a preliminary progress 13 report submitted to the NRC Staff in January, 1978, by 14 International Structural Engineers, Inc. (ISE). ISE was 15 under a consulting contract with the NRC to study contain-16 ment buckling analysis. The preliminary report included a 17 number of preliminary observations which were cited by 18 Mr. Doherty as criticisms of the present predictive methods 19 used for buckling evaluation of containment vessels. ISE's 20 final report was published as NUREG/CR-0793, "Buckling 21 Criteria and Application of Criteria to Design of Steel 22 Containment Shell" (May, 1979). 23

Q. Would you discuss each of the observations made in the consultant's preliminary report which Mr. Doherty cites? A. Those preliminary observations as paraphrased and cited by Mr. Doherty in his contention are quoted and responded to in the following four paragraphs:

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 "Adequate Experimental data for determining design criteria did not exist."

4 Over the past decade a systematic collection has been 5 made by CBI of several hundred technical papers known to 6 contain experimental data on shell buckling. These tests 7 include stiffened and unstiffened shells subjected to a 8 variety of loads or loading combinations. Several of these 9 tests have been performed on models fabricated with procedures 10 representative of those used on containment vessels.

The final consultant's report recognized the fact that 11 adequate experimental data does exist for shells subjected 12 to axisymmetric static loadings. The concern seemed to 13 remain that there may be a lack of data for shells subjected 14 to dynamic asymmetric loadings. This concern will be conserva-15 tively accounted for in the methods employed in design and 16 analysis of ACNGS containment vessel. The specified dynamic 17 loadings will be applied to a mathematical model of the 18 vessel. A shells of revolution program having dynamic 19 analysis capabilities will be used. The resulting stresses, 20 which include the effects of dynamic amplification factors, 21 will then be used as equivalent static stresses for buckling 22 evaluation of the vessel. 23

The asymmetric stress effects are also conservatively treated by applying the maximum stress around the entire azimuth as an axisymmetric (uniform) stress. The final consultants' report recommends this procedure as a conservative approach.

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 "Computer programs for determining buckling loads do not predict experimental buckling results very well."

It is well recognized that the results of computer pro-4 grams based upon classical theory must be modified to predict 5 the buckling capacity of imperfect shells. For the ACNGS 6 vessel, the classical buckling values are reduced by knockdown 7 and plasticity reduction factors, which conservatively 8 account for the difference between the theoretical elastic 9 buckling value for a perfect shell and the critical buckling 10 capacity of a fabricated shell. 11

Both the preliminary and the final consultants' reports endorsed this approach as the preferred method of arriving at the critical buckling loads.

3. "That the ASME Section III Buckling Criteria
Regulatory Guide 1.57, NE-3224 (sic), permits designers to
select the method which yields a buckling stress which is
least conservative."

The classical linear buckling analysis with reductions 19 based on test results, which is the buckling evaluation 20 method used for ACNGS vessel, is the method preferred and 21 recommended by the consultants. This approach, outlined in 22 previous paragraphs, is the most widely used approach for 23 shell buckling evaluation. Applicant does not intend to 24 perform any buckling evaluation for the ACNGS vessel using 25 either of the other two methods permitted. 26

27 4. "Until more test data is obtained to study the
28 effects of imperfections, asymmetric loading, load interaction,

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dynamic and nonlinear effects, a conservative factor of safety such as 3 should be used."

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The final consultants' report recognized that imper-4 fections, asymmetric loadings, load interactions, dynamic 5 loadings, and nonlinear effects can all be treated in a 6 conservative manner, and that a safety factor of 2.0 will be 7 adequate. As the final consultants' report states, "It is 8 felt that a safety factor of 2 is sufficient to achieve a 9 conservative design for all states of stress, if applied to 10 reduction factors obtained as the minimum of experimentally 11 obtained data." This recommendation of the consultants' 12 Report is consistent with the buckling criteria of the ASME 13 Code Case N-284, the requirements of which will be met for 14 this vessel. 15

Q. Would you summarize your opinions concerning Mr.Doherty's contention?

The four (4) observations cited by Mr. Doherty's 18 A . contention have either been superceded in whole or in part 19 by their own authors in the final consultant's report to the 20 NRC (NUREG/CR-0793, May, 1979) or they are well accounted 21 for in the design of the ACNGS containment vessel. The 22 method of analysis employed for the design of the ACNGS 23 containment vessel will result in a conservative prediction of 24 stresses and the buckling evaluation method employed will 25 produce a safe and conservative design. 26

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1	Exhibit KM-1			
2	EDUCATION AND PROFESSIONAL QUALIFICATIONS			
3	KAMRAN MOKHTARIAN			
4	RESIDENCE: BUSINESS:			
5	442 Claremont Court Chigago Bridge & Iron Co.			
6	Downers Grove, Illinois 60516 800 Jorie Blvd.			
7	Oak Brook, Illinois 60521			
8	EDUCATION:			
9	B.S. Degree in Civil Engineering, Cleveland State University,			
10	1963			
11	M.S. Degree in Structural Mechanics, Northwestern			
12	University, 1964			
13	Graduate level courses at Illinois Institute of Technology			
14	EXPERIENCE:			
15	Employed by Chicago Bridge & Iron Co. from 1964 to present.			
16	August 1964-August 1965 - Design Engineer: Working on design			
17	of vacuum chambers and pressure			
18	vessels.			
19	August 1965-June 1966 - Field Engineer: Working on fab-			
20	rication and construction of tanks			
21	and vessels in an oil refinery.			
22	June 1966-August 1967 - Design Engineer: Working on design			
23	and analysis of nuclear reactor			
24	vessels.			
25	August 1967-May 1972 - Group Leader: Having responsibility			
26	for stress analysis of nuclear			
27	reactor vessels and preparation			
28	of ASME Code Stress Reports.			

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1	May 1972-Sept. 1975	- Supervisor of Stress Analysis:
2		Having responsibility for complete
3		design and analysis of nuclear
4		structures. Supervising groups
5		of engineers performing heat
6		transfer analysis, fatigue and
7		fracture analysis, shell and
8		finite element analysis, and
9		buckling analysis. Reviewing
10		and certifying complete Code
11		design and stress reports.
12	Sept. 1975-July 1977	- Project Engineer: Having overall
13		engineering responsibility for
14		design and analysis of the
15		containment vessel for the Clinch
16	선생각 상태생활 영감, 최	River Breeder Reactor Project.
17		Helped develop buckling criteria
18		to be used for the design of that
19		vessel.
20	July 1977-To Date	- Design Supervisor: Having respon-
21		sibility for design of various
22		nuclear structures. Supervising
23	영양 영양 방송 모양 가지?	groups of engineers working on
24		design and analysis of various
25		containment vessels. Helped with
26		developing buckling criteria to
27		be used for design of Mark III
28		containment vessels. Helped with

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1	the development of and authored
2	portions of the ASME Code Case
3	N-284, titled "Metal Containment
4	Shell Buckling Design Methods".
5	PROFESSIONAL REGISTRATION:
6	Registered Professional Engineer in State of Ohio
7	HONOR SOCIETIES:
8	Tau Beta Pi
9	Pi Mu Epsilon
10	PUBLICATIONS:
11	"Hotspot Flexure of Plate on Circular Support",
12	Journal of the Engineering Mechanics Division of
13	ASCE, June 1968
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