

ACNGS-PSAR

HOUSTON LIGHTING & POWER COMPANY  
 ALLENS CREEK NUCLEAR GENERATING STATION - UNIT NO. 1  
 PRELIMINARY SAFETY ANALYSIS REPORT  
 AMENDMENT NO. 58  
 INSTRUCTION SHEET

This amendment contains information pertaining to PSAR Update. Each revised page bears the notation Am. No. 58, (5/81) at the bottom of the page. Vertical bars with the number 58 representing Amendment No. 58 have been used in the margins of the revised pages to indicate the location of the revision on the page.

The following page removals and insertions should be made to incorporate Amendment No. 58 into the PSAR.

REMOVE  
(EXISTING PAGES)

Chapter 2

25\*  
 48\*  
  
 2.5-M7  
 2.5-M8  
 -  
 -  
 FM-1  
 FM-2  
 -

Chapter 3

1\*  
 2\*  
 5\*  
 16\*  
  
 xxxii  
 xxxviii  
  
 3.5-3  
 3.5-5  
 -  
 3.5-9  
 3.5-10  
 3.5-11  
 -  
 3.5-16a  
 3.5-18  
 3.5-19  
 3.5-20  
 F3.5-2

INSERT  
(AMENDMENT NO. 58 PAGES)

Chapter 2

25\*  
 48\*  
  
 2.5-M7  
 2.5-M8  
 2.5-M8a  
 2.5-M14  
 FM-1  
 FM-2  
 FM-2A

Chapter 3

1\*  
 2\*  
 5\*  
 16\*  
  
 xxxii  
 xxxviii  
  
 3.5-3  
 3.5-5  
 3.5-5a  
 3.5-9  
 3.5-10  
 3.5-11  
 3.5-11a  
 3.5-16a  
 3.5-18  
 3.5-19  
 3.5-20  
 F3.5-2

REMOVE  
(EXISTING PAGES)

Chapter 6

1\*  
2\*  
3\*  
4\*  
5\*  
6\*  
6a\*  
7\*  
8\*  
9\*  
10\*  
10a\*  
11\*  
12\*  
13\*  
14\*  
i  
ix  
xiii  
-  
6.2-111  
6.2-112  
F6.2-26a thru n  
F6.2-27a  
F6.2-27b

Chapter 7

1\*  
1a\*

xxi  
xxxix

Chapter 9

1\*  
3\*  
9\*

9.2-15  
9.2-15a  
9.2-18  
-  
F9.2-13  
F9.2-14

Chapter 12

1\*  
2\*  
2a\*

iiia

Chapter 13

1\*

i  
iv

INSERT  
(AMENDMENT NO. 58 PAGES)

Chapter 6

1\*  
2\*  
3\*  
4\*  
5\*  
6\*  
6a\*  
7\*  
8\*  
9\*  
10\*  
10a\*  
11\*  
12\*  
13\*  
14\*  
i  
ix  
xiii  
xiiia  
6.2-111  
6.2-112  
-  
F6.2-27a & 27b  
-

Chapter 7

1\*  
1a\*

xxi  
xxxix

Chapter 9

1\*  
3\*  
9\*

9.2-15  
9.2-15a  
9.2-18  
9.2-18oa  
F9.2-13  
F9.2-14

Chapter 12

1\*  
2\*  
2a\*

iiia

Chapter 13

1\*

i  
iv

REMOVE  
(EXISTING PAGES)

Chapter 15

1\*  
9\*

APPENDIX C

1\*  
2\*

iv

C1.52-1

INSERT  
(AMENDMENT NO. 58 PAGES)

Chapter 15

1\*  
9\*

APPENDIX C

1\*  
2\*

iv

C1.52-1

## ACNGS-PSAR

EFFECTIVE PAGES LISTING (Cont'd)  
CHAPTER 2

<u>Page No.</u>	<u>Amendment No.</u>
i (Appendix L, Section 2.5)	32
ii	32
iii	32
iv	32
v	32
vi	32
vii	32
1-1	32
1-2	32
2-1	32
3-1	32
4-1	32
4-2	32
5-1	32
5-2	32
5-3	32
5-4	32
6-1	32
7-1	32
7-2	32
7-3	32
7-4	32
7-5	32
7-6	32
2.5-M1 (Appendix M, Section 2.5)	50
2.5-M2	50
2.5-M3	50
2.5-M4	50
2.5-M5	50
2.5-M6	50
2.5-M7	58
2.5-M8	58
2.5-M8a	58
2.5-M9	50
2.5-M10	50
2.5-M11	50
2.5-M12	50
2.5-M13	50
2.5-M14	58



EFFECTIVE FIGURES LISTING (Cont'd)  
CHAPTER 2  
SITE CHARACTERISTICS

<u>Figure No.</u>	<u>Amendment No.</u>
18	32
19	32
20	32
21	32
22	32
23	32
24	32
M1 (Appendix M, Section 2.5)	58
M2	58
M2A	58
M3a	50
M3b	50
M4	50
M5a	50
M5c	50
M6	50
M7a	50
M7b	50
M8	50
M9a	50
M9b	50
M9c	50
M10	50
M11	50
M12	50
M13	50
M14	50
M15	50
M16	50
M17	50
M18	50
M19	50
M20	50
M21	50
M22	50
M23	50
M24	50
M25	50
M26	50
M27	50
M28	50
M29	50
M30	50
M31	50
M32	50
M33	50
M34	50
M35 & 36	50

applicable, the horizontal (0.1g) and vertical (0.067g) components of the design basis earthquake are input.

To find the worst possible radius and center of rotation yielding the circle with the lowest factor of safety, a search routine is built into the program by which a trial center of rotation is selected. The program will investigate different radii from that center of rotation computing and recording the safety factor for each radius. It then moves the center of rotation at a prescribed increment to a different trial location and the above process is repeated until the lowest safety factor is reached.

The simplified Bishop solution yields results that are conservative in that shear resistance between slices, which would tend to raise the factor of safety against sliding, is neglected. When the simplified Bishop solution is used to compute a factor of safety under dynamic loading additional conservatism is built into the program in that the computed safety factor is calculated assuming the components of the design earthquake acceleration act only in one direction, neglecting any back and forth motion, and the magnitude of the acceleration of the design earthquake is taken to be a constant over the entire slope for an infinite length of time.

In performing the sliding wedge method the Ebasco computer program was also used. The sliding wedge method consists of an active wedge being mobilized against a neutral horizontal block and a passive resisting wedge. The factor of safety is calculated as the ratio of the sum of the resisting forces in the horizontal direction to the sum of the driving forces in the horizontal direction. In applying the sliding wedge method to the two cross-sections the input data and search routine is similar to that of the slip circle analysis previously discussed. This method also includes a seismic loading in the analyses. This was done by including the product of the weights of the wedges and the neutral block with the horizontal acceleration factor of 0.1g. This force was then considered to act in the direction of the postulated slide as a driving force. The vertical component of the seismic loading is also incorporated into the solution tending to reduce frictional resistance between the sliding wedges. This vertical seismic force is computed as the product of the weights of the neutral block and the wedges with the vertical acceleration factor of 0.067g.

The results of each of these analyses are presented on the tables on Figure M2. In all cases the actual safety factor exceeds the recommended minimum safety factor from Table M5, indicating that the slopes are safe.

The above described detailed investigation has accurately established the soil conditions in the area of the ultimate heat sink at Allens Creek. The continuous sampling in the upper soils and careful undisturbed sampling of clays and sand establishes a sound basis for the selection of lower bound strength samples. Selection of design strength parameters incorporated the use of lower bound strength parameters from the test results, using very conservative test procedures. Results of the analyses indicated

50  
361.4

58

satisfactory safety factors. Reflected in the analyses are the changes required to obtain the required safety factors. In order to maintain the 1 vertical to 3 horizontal slope of the causeway it was necessary to excavate the surface clays from beneath the causeway. Additionally the slopes of the ultimate heat sink basin have been flattened to 1 vertical to 8 horizontal from the original 1 vertical to 3 horizontal. These changes are the result of using the  $\phi = 9^{\circ}$  from the consolidated drained repeated direct shear tests.

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361.4

#### M8 UHS CAUSEWAY SLOPE STABILITY

In accordance with the design parameters and loading criteria outlined in Figure M-2, studies were performed to verify the design and the validity of the slope stability of the UHS Causeway. Directly underneath the Causeway there exists an approximate 50 foot thick layer of dense sand. This dense sand layer will act as a firm foundation and provide adequate support and sufficient stability for the Causeway. An analysis simulating a potential failure was conducted and postulated that the potential failure plane would occur within the recompacted material, i.e. the man-made Causeway. For this failure a minimum safety factor of 1.46 against seismic loading conditions as shown in Table M6 would result.

A parametric study was also performed using the Seismic Wedge Method, considering side friction along the potential failure plane. The failure plane was allowed to cut through 50 feet of dense sand, and to penetrate into the underlying clay layer. This study yielded a minimum safety factor of 1.34 and approximated the results obtained from the Simplified Bishop Method. Either method is considered acceptable and adequate for the Causeway design.

58

It should be noted that the constant excitation forces utilized in both wedge and Simplified Bishop Method slope stability analyses provide an extremely conservative approach in the case of an instantaneously back and forth earthquake motion of 10 second duration. This conservatism was also coupled with an unrealistic large amount of driving mass in the Wedge Method as indicated in Figure M-2. A comparison of these parametric study results is presented in Table M6.

In summary, the slope stability results as presented in Figure M-2 are extremely conservative, since they neglect all side friction forces along the failure planes, are coupled with unrealistically large amounts of driving mass, and neglect the low average soil design parameters. The minimum safety factors for the Causeway design are well within acceptable limits for different loading conditions as established by U.S. Army Corps of Engineers. (See PSAR Section 2.5 Appendix M Table M5.)

Furthermore, in accordance with the NRC Regulatory Guide 1.27, an analysis postulating the failure of the man-made Causeway was performed. A cross section taken at the lake front area was investigated for the potential blockage of the waterway should the Causeway be postulated to fail in this area. The results of the slope stability analysis along with postulated failure plane are presented in Figure M-2a.

The movement of the postulated failure plane was examined using Newmark's method (1) with a safety factor equal to 1, i.e. an impending failure of the Causeway slope. This yielded a conservative soil movement of less than 4 inches. This soil movement would produce a very minor bulging type of deformation which would be experienced in the Causeway along the lake front area (see Figure M-2a). As an additional conservatism and in order to provide a positive stoppage of soil movement, a concrete retaining wing wall structure will be provided at the Causeway/lake front area (see Figure M-1) to assure a continuous passage of cooling water into the fore-bay canal of the Ultimate Heat Sink Intake Structure.

58

Embedded piping and electric cabling within the Causeway embankment has been designed to ensure physical independence and redundancy following postulated failure of the Causeway. Figures M-2 and M-2a contain Causeway cross sections illustrating piping and electric cabling corridors.

(1) "Effects of Earthquake on Dams and Embankment", Fifth Rankine Lecture by N.M. Newmark, Geotechnique, London, England, 1965.

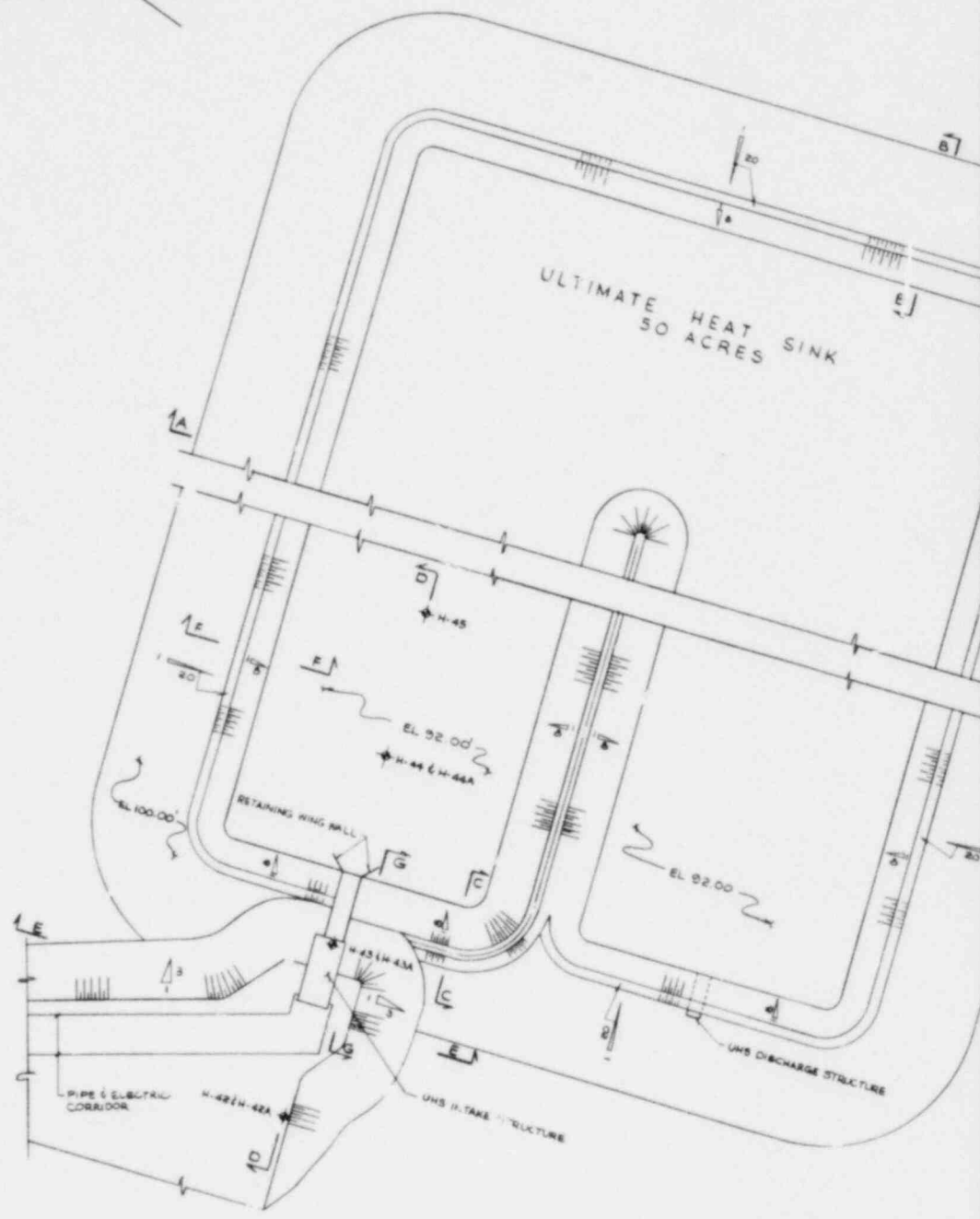
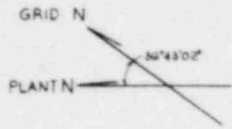
58

TABLE M6

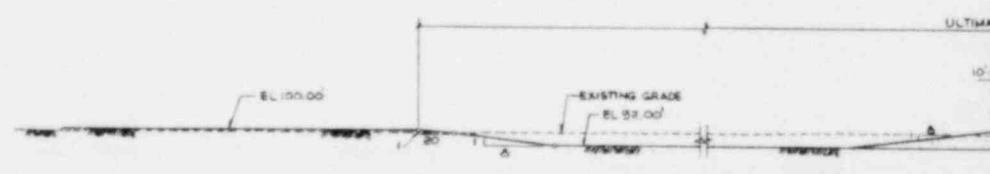
COMPARISON OF THE RESULTS OF THE PARAMETRIC STUDY

<u>Methods and Conditions</u>	<u>Factor of Safety</u>	<u>Remark</u>
Seismic Simplified Bishop Slip Circle	1.35	
Seismic Wedge Method using average low test results	1.46	Failure plane occurs within causeway
Seismic Wedge Method with friction resistance along failure plane	1.34	Failure plane cutting through 50 ft dense sand and penetrating into under clay
Seismic Wedge Method, using average low test result and neglecting friction resistance	1.15	Failure plane cutting through 50 ft dense sand and penetrating into under clay

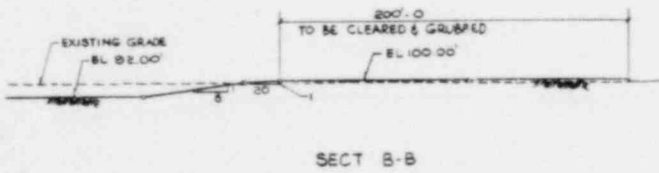
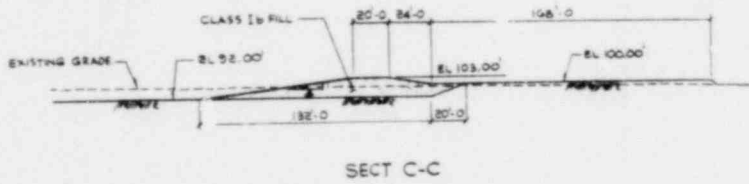
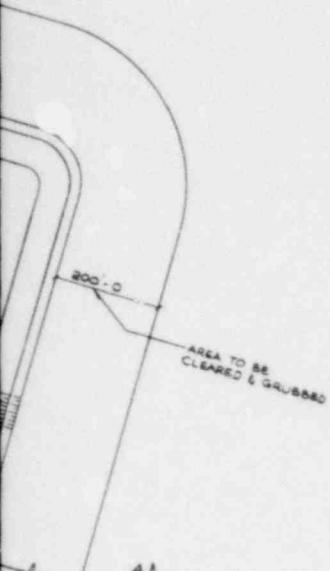
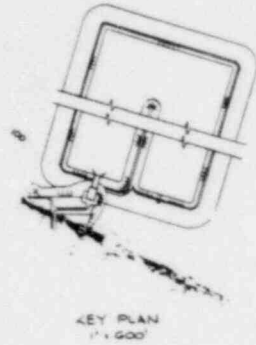
Design parameters:  $C = 1000$  psf for clay,  $\phi = 38^\circ$  for sand  
 Loading Criteria: SSE and full reservoir WL = EL 118 ft



PLAN & BORING LOCATION



POOR ORIGINAL



AM. NO. 58, (5/81)

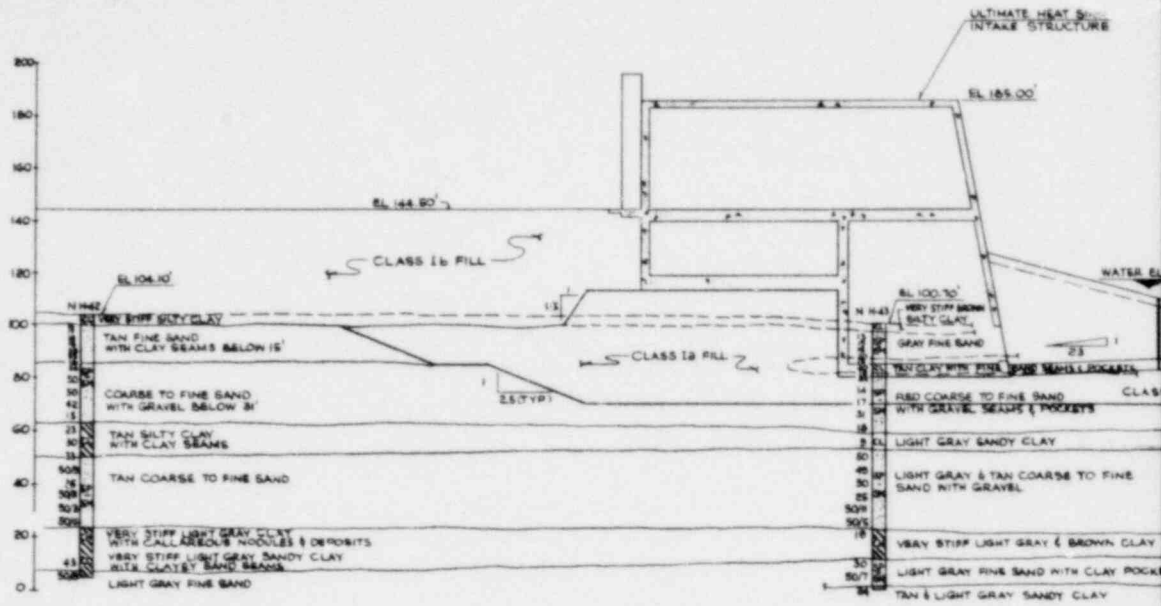
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Allens Creek Nuclear Generating Station  
Unit 1

ULTIMATE HEAT SINK  
PLAN AND SECTIONS

FIGURE M-1

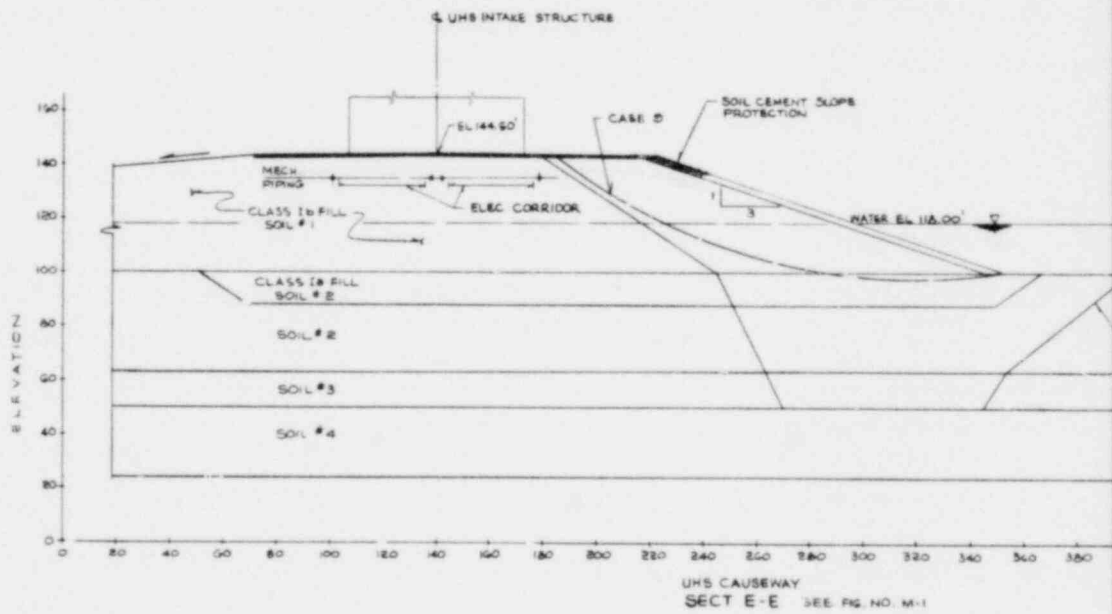
POOR ORIGINAL





SECT D-D

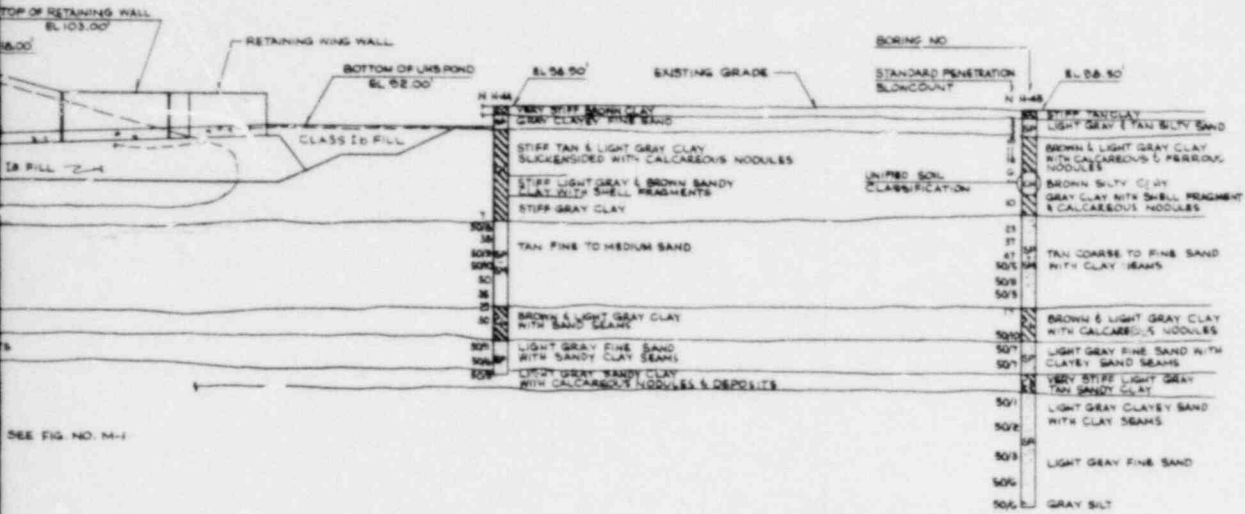
CASE NO.	STRUCTURE	ANALYSIS TYPE	FAILURE PLANE	GROUND WATER LEVEL	SOIL PROPERTIES	SUGGESTED MIN. SAFETY FACTOR	ACTUAL MIN. SAFETY FACTOR
1	UHS CAUSEWAY	STATIC	CIRCLE	NORMAL (EL 118.00')	#1 C=1000PSF, #2 $\phi=38^\circ$ , #3 C=1000PSF, #4 $\phi=38^\circ$	1.5	1.56
2			WEDGE			1.5	1.76
3			CIRCLE		#1 $\phi=38^\circ$ , C=300PSF, #2 $\phi=38^\circ$ , #3 $\phi=38^\circ$ , C=300PSF, #4 $\phi=38^\circ$	1.3	1.52
4			WEDGE			1.3	2.24
5			CIRCLE		#1 $\phi=38^\circ$ , C=300PSF, #2 $\phi=38^\circ$ , #3 $\phi=38^\circ$ , #4 $\phi=38^\circ$	1.25	1.52
6		STATIC	WEDGE	NORMAL (EL 118.00')		1.25	1.52
7		SAFETY DRABDOWN	CIRCLE	NORMAL TO EL 111.00'	#1 C=1000PSF, #2 $\phi=38^\circ$ , #3 C=1000PSF, #4 $\phi=38^\circ$	1.2	2.10
8		SAFETY DRABDOWN	WEDGE	NORMAL TO EL 111.00'		1.2	1.56
9		EARTHQUAKE	CIRCLE	NORMAL	#1 C=1000PSF, #2 $\phi=38^\circ$ , #3 C=1000PSF, #4 $\phi=38^\circ$	1.15	1.35
10	UHS CAUSEWAY	EARTHQUAKE	WEDGE	NORMAL		1.15	1.15



UHS CAUSEWAY  
SECT E-E SEE FIG. NO. M-1

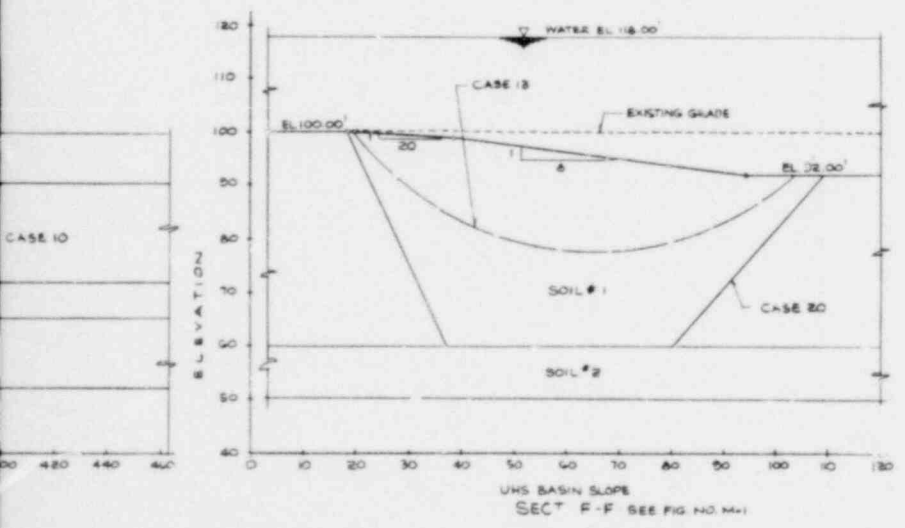
POOR ORIGINAL





SEE FIG. NO. M-1

CASE NO.	STRUCTURE	ANALYSIS TYPE	FAILURE PLANE	GROUND WATER LEVEL	SOIL PROPERTIES	SUGGESTED MIN. SAFETY FACTOR	ACTUAL MIN. SAFETY FACTOR
11	UHS BASIN SLOPE	STATIC	CIRCLE	NORMAL (EL 116.00')	#1 C=1000PSF, #2 2 x 36"	1.5	13.20
12			WEDGE			1.5	12.31
13			CIRCLE		#1 2 x 2', C=300PSF, #2 2 x 36"	1.5	5.38
14			WEDGE			1.5	5.28
15			CIRCLE		#1 2 x 2", #2 2 x 36"	1.25	1.25
16		STATIC	WEDGE	NORMAL (EL 116.00')		1.25	2.25
17		RAPID DRANDOM	CIRCLE	NORMAL TO EL 100.00'	#1 C=1000PSF, #2 2 x 36"	1.2	13.26
18		WIND DRANDOM	WEDGE	NORMAL TO EL 100.00'		1.2	2.37
19		EARTHQUAKE	CIRCLE	NORMAL	#1 C=1000PSF, #2 2 x 36"	1.15	3.52
20	UHS BASIN SLOPE	EARTHQUAKE	WEDGE	NORMAL		1.15	4.29



UHS BASIN SLOPE  
SECT F-F SEE FIG. NO. M-1

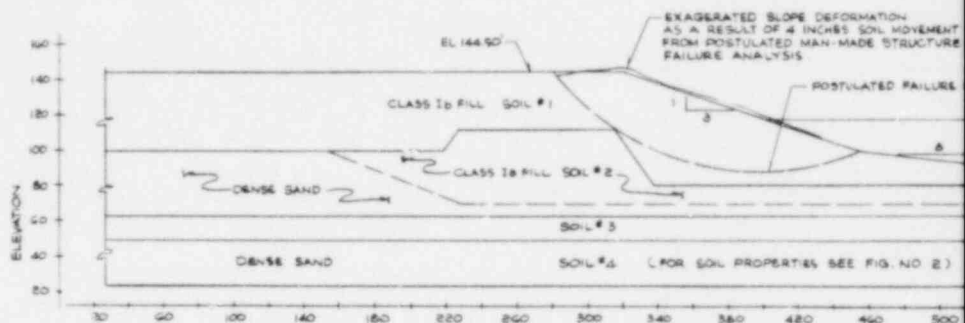
AM. NO. 58, (5/81)

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Allens Creek Nuclear Generating Station  
Unit 1

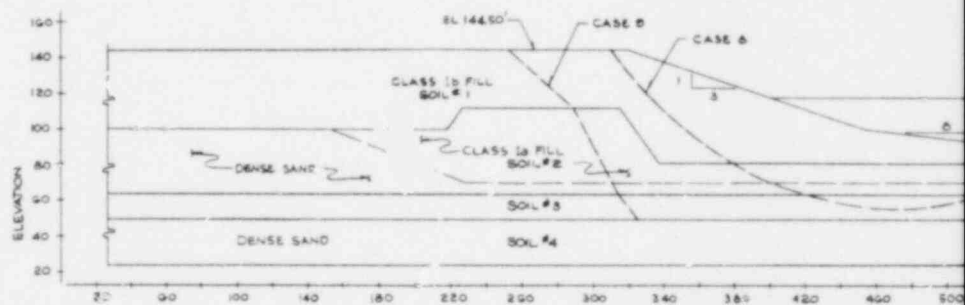
U.H.S. CAUSEWAY - STABILITY ANALYSES  
FIGURE M-2

POOR ORIGINAL

CASE NO.	STRUCTURE	ANALYSIS TYPES	LOADING CRITERIA	GROUND WATER (ELEV)	SOIL PROPERTIES
1	LHS CAUSEWAY	SB CIRCLE	STATIC UNDRAINED	118.00'	$\phi = 1000$ PSF, $\phi = 35^\circ$
2		SB CIRCLE	STATIC UNDRAINED	110.00'	"
3		WEDGE	STATIC UNDRAINED	118.00'	"
4		SB CIRCLE	STATIC DRAINED	118.00'	$\phi = 217$ , $C = 300$ PSF, $\phi = 30^\circ$ , $\phi = 35^\circ$
5		WEDGE	STATIC DRAINED	118.00'	"
6		SB CIRCLE	STATIC RESIDUAL	118.00'	$\phi = 187$ , $C = 300$ PSF, $\phi = 30^\circ$ , $\phi = 35^\circ$
7		SB CIRCLE	STATIC RESIDUAL	118.00'	"
8		SB CIRCLE	SEISMIC UNDRAINED	118.00'	$\phi = 1000$ PSF, $\phi = 35^\circ$
9	LHS CAUSEWAY	WEDGE	SEISMIC UNDRAINED	118.00'	"



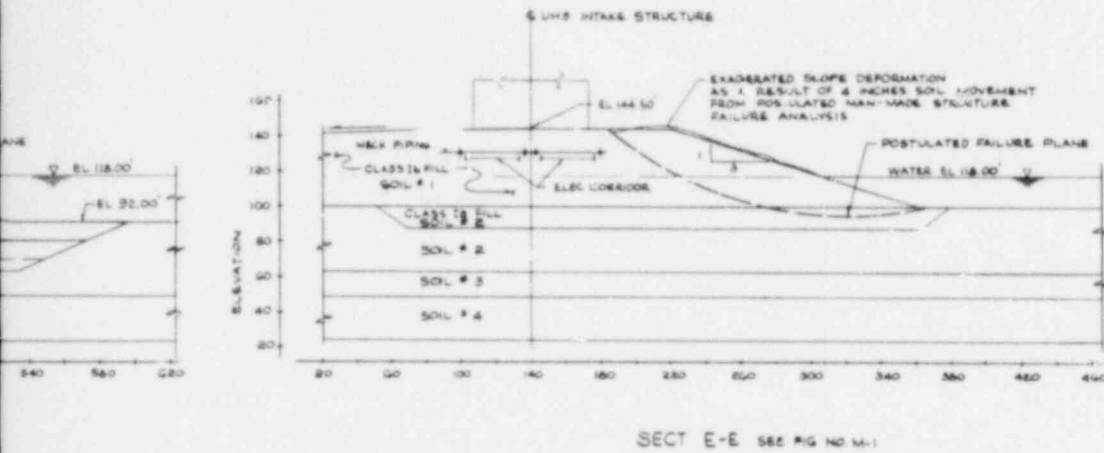
SECT G-G SEE FIG NO M-1



SECT G-G SEE FIG NO M-1

POOR ORIGINAL

PIES	SUGGESTED MIN. SAFETY FACTOR	ACTUAL MIN. SAFETY FACTOR	REMARKS
1000 PSF #4 2'-38"	1.5	1.75	FUL. RESERVOIR
" " " "	1.5	1.75	RAPID DRAWDOWN
" " " "	1.5	2.15	
7000 PSF #4 2'-38"	1.5	1.77	
" " " "	1.5	2.10	
1'-0" #4 2'-38"	1.5	1.65	SLICKENSIDED CLAY CONDITION
" " " "	1.5	1.75	SLICKENSIDED CLAY CONDITION
1000 PSF #4 2'-38"	1.15	1.51	EARTH-LAKE CONDITION
" " " "	1.15	1.20	EARTH-LAKE CONDITION



SECT E-E SEE FIG NO M-1

AM. NO. 58, (5/81)

HOUSTON LIGHTING & POWER COMPANY  
Allens Creek Nuclear Generating Station  
Unit 1

U.H.S. CAUSEWAY - STABILITY ANALYSES  
FIGURE M-2A

POOR ORIGINAL

LIST OF EFFECTIVE PAGES  
CHAPTER 3

## DESIGN OF STRUCTURES, COMPONENTS, EQUIPMENT AND SYSTEMS

<u>Page</u>	<u>Amendment</u>
1*	58
2*	58
3*	56
4*	57
5*	58
6*	48
7*	49
8*	57
9*	57
10*	56
11*	57
12*	56
12c	56
13*	42
14*	47
15*	48
16*	58
16a*	39
17*	54
18*	57
i	35
ii	35
iii	35
iv	35
v	35
vi	35
vii	35
viii	35
ix	35
x	35
xi	35
xii	35
xiii	37
xiv	35
xv	35
xvi	44
xvii	44
xviii	44
xix	48
xx	35
xxi	44
xxii	35
xxiii	35
xxiv	35
xxv	35

\* Effective Pages/Figures Listing

## ACNGS- PSAR

LIST OF EFFECTIVE PAGES (Cont'd)  
CHAPTER 3

<u>Page</u>	<u>Amendment</u>
xxvi	37
xxvii	41
xxviii	56
xxviii a	56
xxviii b	56
xxix	35
xxx	35
xxxi	35
xxxii	58
xxxiii	48
xxxiii a	56
xxxiv	56
xxxv	39
xxxvi	44
xxxvii	48
xxxvii a	48
xxxviii	58
xxxix	56
3.1-1	-
3.1-2	56
3.1-3	-
3.1-4	-
3.1-5	35
3.1-6	35
3.1-7	35
3.1-8	35
3.1-9	56
3.1-10	35
3.1-11	35
3.1-12	56
3.1-13	-
3.1-14	-
3.1-15	-
3.1-16	35
3.1-17	35
3.1-18	35
3.1-19	56
3.1-20	35
3.1-21	56
3.1-22	56
3.1-22 a	35
3.1-22 b	35
3.1-23	56
3.1-24	56
3.1-25	35
3.1-26	56
3.1-27	56
3.1-28	56
3.1-29	56

EFFECTIVE PAGES LISTING  
CHAPTER 3DESIGN OF STRUCTURES, COMPONENTS, EQUIPMENT AND SYSTEMS

<u>Page</u>	<u>Amendment</u>
3.3-1	35
3.3-2	35
3.3-3	48
3.3-4	35
3.3-5	35
3.3-6	35
3.3-7	35
3.3-8	35
3.4-1	45
3.4-1a	35
3.4-2	35
3.4-3	35
3.4-4	37
3.4-5	35
3.5-1	35
3.5-2	48
3.5-2a	35
3.5-3	58
3.5-4	35
3.5-5	58
3.5-5a	58
3.5-6	37
3.5-7	35
3.5-8	35
3.5-9	58
3.5-10	58
3.5-11	58
3.5-11a	58
3.5-12	35
3.5-13	37
3.5-14	35
3.5-15	35
3.5-15a	35
3.5-15b	44
3.5-16	49
3.5-16a	58
3.5-17	39
3.5-18	58
3.5-19	58
3.5-20	58
3.5-21	35
3.6-1	35
3.6-2	35
3.6-3	35
3.6-3a	35
3.6-3b	35
3.6-3c	35
3.6-3d	35

## EFFECTIVE FIGURE LIST\*

## CHAPTER 3

DESIGN OF STRUCTURES, COMPONENTS, EQUIPMENT AND SYSTEMS

<u>Figure No.</u>	<u>Amendment No.</u>
3.2-1	-
3.2-2	26
3.4-1	13
3.4-2	37
3.4-3	37
3.4-4	37
3.4-5 (Deleted)	37
3.4-6	37
3.5-1	37
3.5-2	58
3.5-3	1
3.5-4	1
3.6-1	35
3.6-2	35
3.6-3a	35
3.6-3b (Both on same page)	35
3.6-4	35
3.6-5	35
3.6-6	35
3.6-7	35
3.6-8	35
3.6-8a	35
3.6-9	35
3.6-10	35
3.6-11	35
3.6-12	42
3.6-13	42
3.7-1	35
3.7-2	35
3.7-3	35
3.7-4	35
3.7-5	35
3.7-6	35
3.7-7	35
3.7-8	35
3.7-9	35
3.7-10	35
3.7-11	35
3.7-12	35
3.7-13	35
3.7-14	35
3.7-15	35
3.7-16	35
3.7-17	44
3.7-17a	44
3.7-18	35
3.7-19 (Amendment No. not shown on page)	35
3.7-20	35

\* All Figures whether labelled "Unit 1" or "Units 1 & 2" are to be considered applicable to Unit No. 1.

LIST OF TABLES

<u>Table</u>	<u>Title</u>	<u>Page</u>
3.2-1	Equipment Classification	3.2-15
3.2-2	Design Requirements for Quality Group A and Safety Class I Systems, Structures, and Components	3.2-35
3.2-3	Design Requirements for Quality Group B and Safety Class 2 Systems, Structures, and Components	3.2-36
3.2-4	Design Requirements for Safety Class 2 and 3 Systems and Components	3.2-37
3.2-5	Design Requirements for Quality Group C and Safety Class 3 Systems, Structures and Components	3.2-38
3.2-6	NOTES FOR TABLES 3.2-2, 3.2-3, 3.2-4, 3.2-5	3.2-39
3.2-7	Correlation of Quality Group Designations with Industry Codes and Standards for Mechanical Components	3.2-40
3.2-8	Conditions of Design for Safety Class 1, 2 and 3 components	3.2-41
3.2-9	Summary of Safety Class Design Requirements	3.2-43
3.2-10	Active Valves in Seismic Category I Systems	3.2-44
3.3-1	Vertical Wind Speed and Loading Distribution for Plant Structures	3.3-7
3.4-1	Flood Protection Measures	3.4-4
3.4-2	Ultimate Heat Sink Intake Structure Net Hydrostatic and Hydrodynamic forces	3.4-5
3.5-1	Structures Designed for Missiles and Missile Types	3.5-17
3.5-2	Intentionally Deleted	3.5-18
3.5-3	Turbine Missile Characteristics (Typical, for 38" Wheel)	3.5-19
3.5-4	Probability of a Turbine Missile Impacting Plant Safety Related Structures	3.5-20
3.5-5	Characteristics of Tornado Generated Missiles	3.5-21



ACNGS-PSAR

LIST OF FIGURES (CONT'D)

<u>Figure</u>	<u>Title</u>	
3.8-1	Containment Vessel Structural Features	
3.8-2	Deleted	54
3.8-3	Reactor Containment Building Internal Structures Base Details	
3.8-4	Reactor Building Piping Penetrations	
3.8-5	Reactor Building Steel Plate RPV Pedestal	54
3.8-6	Reactor Building Reactor Shield Wall	
3.8-7	Typical Equipment Foundation	
3.8-8	Typical Reinforcement Detail	
3.8-9	Drywell Base Detail	
3.8-11	Reactor Building Dome - M&R	54
3.8-12	Containment Vacuum Breaker $A\sqrt{K}$ vs. Maximum Containment Negative Pressure	
3.8-13	Small Line Break Inside the Containment	
3.8-14	Containment Response After Drywell and Containment Vacuum Breaker Initiation	
3.8-15	Drywell Negative Pressure vs. Time for Steam Condensation Following Small Primary System	
3.8-16	Relative Effects of Heat Sinks and Spray Depressurization	
3.8-17	Inadvertent Spray Activation	
3.8-18	Three Foot Containment Penetration Dedicated For Degraded Core Rule Making	58
3.9-1	RPV and Internals Vertical Dynamic Model	
3.9-2	The Amplification Factor $\mu$ as a Function of the Frequency Ratio $\pi$ for Various Amounts of Viscous Damping	

from turbinning of the driven end of the equipment due to blowdown of the system pressure upon rupture of the system pressure boundary.

The most substantial piece of rotating equipment is the recirculation pump and motor which, in the event of a major recirculation line break, and under certain system blowdown conditions, can theoretically reach overspeed beyond practical design limitations and result in ejection of various parts of the pump and motor. This hypothetical situation is currently the topic of discussions between GE and the NRC. The Applicant will implement the generic resolution of these discussions.

49

### 3.5.2.2 Turbine Missiles

#### 3.5.2.2.1 Introduction

The potential for damage to safety related structures, systems and components due to turbine failure has been evaluated to determine whether additional protection, beyond that inherently provided by plant building orientation and existing structural shielding, need be provided to further reduce the probability of damage.

The probability of damage was calculated for the vital plant structures by evaluating the product of the probability of missile generation, the probability of impact on the structure and the probability of damage to the structures.

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The evaluation of the individual probability components and a summary of the overall damage probability is discussed in the following sections.

testing techniques, are now better able to discover surface and internal defects. In addition, a thorough inspection program will be used to inspect the critical bore and keyway regions to minimize the possibility of stress corrosion cracking. An ultrasonic test has been developed to detect stress corrosion cracks in these regions well before cracks would grow to critical size necessary to cause turbine wheel failure. Laboratory investigation has revealed some of the basic relationships among structure strength, material strength, FATT, and defect size and location, so that the reliability of the rotor as a structure has been significantly improved during the past few years.

58

New starting and loading equipment and instructions reduce the severity of surface and bore thermal cycles incurred during service. The improvements include: better temperature sensors; better guidance for station operators in the control of speed, acceleration, and loading rates to minimize rotor stresses.

Progress in design, better materials and quality control, more rigorous acceptance criteria, and improved machine operation have substantially reduced the likelihood of burst failures of turbine-generator rotors operating near rated speed.

d) Turbine-Generator Overspeed Protection

The improvements of rotor quality discussed above reduce the chance of failures at operating speed, but they do tend to increase the hazard level associated with unlimited overspeed, because of the greater missile energy associated with higher bursting speed. Therefore, it is pertinent to examine the Turbine Overspeed Protection Systems. For condensing units, the devices to control the flow of steam into the turbine are discussed in the following paragraphs.

e) Main and Secondary Steam Inlets

Main and secondary steam inlets have valves in series. These valves are:

- 1) Control valves, or throttle valves, controlled by the speed governor and tripped closed by emergency governor and backup overspeed trip. | 35(U)
- 2) Stop valves or trip valves, actuated by the emergency governor and backup overspeed trip. | 35(U)

Emergency main stop valves of the steam sealed design have been used on General Electric steam turbines of 10,000 KW and larger since 1948. More than 650 turbines have been shipped and placed in service during this period, and there has been no report of the main stop valve failing to close when required to protect the turbine. Impending sticking is disclosed by the full closed test feature so that a planned shutdown could be made to permit the necessary correction. Such correction almost always requires removal of the

| 17

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ACNGS-PSAR

oxide layer that builds up in the stem and bushing, which would not occur on a low temperature nuclear application.

Steam-driven auxiliary turbines, like the main units, include two complete lines of defense: control and stop valves, speed and emergency governors against destructive overspeed.

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have less stored energy than corresponding parts of the low-pressure turbine. These turbines are totally enclosed in a shielded compartment on upper mezzanine floor of the Turbine Building.

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4) High-pressure turbine rotor

This rotor would not be expected to fail at runaway speed. But, even if failure did occur, the fragments should be retained by the heavy-section, bolted high-pressure shells.

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5) Couplings

Couplings are designed to withstand overspeed higher than the maximum speed at runaway.

The following components could produce high energy missiles:

1) Low-pressure turbine wheels

The wheel capable of producing the most dangerous missile is the last stage. Using the analysis techniques described in Reference 3.5-3, it has been shown that a 120 degree fragment is the most dangerous in terms of a concrete slab that can be perforated after leaving the turbine casing.

2) Bucket vane

The last stage bucket is both the heaviest and most energetic of the vanes capable of escape from the turbine casing. Favorably oriented, i.e., head-on, the vane could conceivably penetrate approximately 10 inches of steel. Such orientation is unlikely, as the tip of the bucket would quickly strike the outer diaphragm ring tangentially. Interference with the other buckets could also be expected. In the ensuing tangle, the inner casing and hood structure would probably contain the blade.

At worst, the vane could ricochet and escape through the 1-1/4 inch-thick plate toward the end of the hood. It is judged that in any case no more than half of the initial energy is retained.

i) Turbine Missiles - Probability Analysis

The present analysis utilizes the historical turbine failure probability value of  $4 \times 10^{-5}$  per year per unit for destructive overspeed failures (see Reference 3.5-22). Analyses have indicated that for destructive overspeed failure, the initial energy imparted to post-lated last stage turbine missiles is  $4 \times 10^6$  ft-lb<sub>f</sub>. The missile energy outside the turbine casing is  $20.5 \times 10^6$  ft-lb<sub>f</sub> and the turbine casing absorbs  $20.5 \times 10^6$  ft-lb<sub>f</sub> (see Table 3.5-3).

58

Studies of design overspeed failure when applied to a turbine having 38" last stage buckets, indicate that the initial energy imparted to a postulated turbine missile is approximately  $16 \times 10^6$  ft-lb<sub>f</sub>. This energy is not sufficient to penetrate the turbine casing, and consequently, all missiles associated with a design overspeed failure would be contained. Alternatively, if missile should perforate the turbine casing, their residual energy would be so low that they could not damage any safety-related structure.

58

### 3.5.2.2.3 Probability of Impact with Safety Related Structures

Figure 3.5-1 shows the relative location of low pressure elements of the turbines and the plant structures. Missiles may be ejected at any angle of the 360 degrees about the turbine axis. The missile ejection angles and directions are illustrated in Figure 3.5-2.

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Tests have indicated (Reference 3.5-5) that deflection angles ( $\theta_2$ ) of turbine generated missiles will be close to zero ( $\pm 5$  degrees) for interior discs and 0 to 25 degrees for the last stage disc. A uniform probability distribution is assumed within this range of angles. To calculate the impact probability it was also assumed that a missile would be ejected with equal probability in the 360 degrees around the rotor axis ( $\theta_1$ ).

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Table 3.5-3 shows the characteristics of a typical turbine missile from the last stage wheel.

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#### a) High Trajectory Turbine Missile Probabilities

A convenient means for estimating the probability of strikes from high trajectory missiles lies in calculating the overall extent of the region which the missiles can reach. Because of the plant arrangement, the striking of all the vital plant structures with high trajectory missiles requires that these missiles be ejected at angles bounded by  $85 < \theta < 90$ .

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Figure 3.5-3, which is based on information contained in References 3.5-5 and 3.5-16, is a map of the x-y ground plane locations into which the missile indicated in Table 3.5-3 would fall. The turbine wheel is shown at the intersection of the x and y-axis.

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$\theta_1$  is the angle between the velocity vectors' projection onto the Y-Z plane (the plane of the turbine wheel) and the Y direction measured in the Y-Z plane.

$\theta_2$  is the angle between the velocity vectors' projection onto the Y-X plane (the horizontal plane) and the Y direction measured in the Y-X plane.

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$\Phi$  is the true angle between the velocity vector and the Y-X plane (the horizontal plane) measured in the plane normal to the horizontal plane in which the vector lies.

See Figures 3.5-1 through 3.5-4 for a graphical representation of these angles.

The maximum range achievable neglecting air resistance (air resistance changes range probability distribution but its impact is not significant) is with  $\Phi$  equal to 45 degrees. Locations were calculated by varying  $\theta_1$  from 0 to 90 degrees. A mirror image plot around the x-axis would result for values from 90 to 180 degrees. Values from 180 to 360 degrees would be meaningless, for these missiles are ejected below the ground. Calculations were made for values of  $\theta_2$  from 0 to 25 degrees which encompasses the maximum postulated turbine missile deflection angle.

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37(U)

The curves in the lower half of Figure 3.5-3 show landing zones for missiles released at values of  $\theta_1$  up to 45 degrees. The curves above the y-axis define landing zones for high trajectory missiles, for which  $\theta_1$  lies between 45 and 90 degrees. The two patterns are actually coincident in each of the four quadrants of the x-y plane. The dotted curves and arrowheads in the lower half of Figure 3.5-3 represent an overlay of the high trajectory curves on the low trajectory solid curves. The dotted curves and arrowheads in the upper half represent an overlay of the low trajectory curves on the high trajectory solid curves.

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The significance of this figure lies in the uniform distribution of the angles  $\theta_1$  and  $\theta_2$ . Because of this, and because the lines are calculated at uniform subdivisions of those distributions, each landing zone defined by a pair of  $\theta_1$  lines and a pair of  $\theta_2$  lines has an equal probability of being struck. Further, locations outside of the region bounded by  $r = 6,900$  feet and the appropriate limiting value of  $\theta_2$  cannot be reached by the missile.

37(U)

All of the vital plant structures lie within the region bounded by  $85 < \theta_1 < 90$  and  $0 < \theta_2 < 5$  degrees. Thus the probability of striking any particular plant structure is only related to the plane (horizontal) area of the structure, and is given by the ratio of that area to the area of the region which can be struck by the missile. The latter in turn is conservatively estimated to be the area of half the ellipse having a major axis equal to the maximum range of the missile ejected with  $\theta_2 = 0$  degree and a minor axis with  $\theta_1 = 90$  degrees and any specified  $\theta_2$ . The probability of impact from high trajectory missiles on the safety related structures is given in Table 3.5-4 together with the aggregate total damage probability.

58

b) Low Trajectory Missiles

Due to the plant arrangement, orientation of the turbine generators, and the relative elevations of the turbine operating deck and the other structures, no safety related structure is exposed to impact damage by low trajectory missiles. As shown on Figure 3.5-1 only the Radwaste Building is located within the 25 degrees deflection limit.

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The Radwaste Building is located below the turbine operating deck level. In order to impact directly on the roof of this structure,



it would be necessary to penetrate the operating floor at an impact angle of five to fourteen degrees from horizontal. The operating floor is composed of reinforced concrete three feet thick. (See Figure 1.2-22). At these angles of impact, the missile would not penetrate the operating floor.

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To impact the walls of the Radwaste Building adjacent to the Turbine Building, it would be necessary for the missile to penetrate the reinforced concrete turbine pedestal and several reinforced concrete internal walls at the mezzanine level. Therefore, the missile would not have sufficient energy to reach these walls.

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#### Summary and Conclusion

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The high trajectory turbine missiles are characterized by their nearly vertical trajectories. The total damage probability of a high trajectory turbine missile striking the safety related structures is less than  $10^{-7}$  per unit year as listed in Table 3.5-4. In addition, the vulnerable safety related equipment area which is exposed to the potential turbine missile is redundant and physically well separated. Consequently the risk from high trajectory turbine missiles is insignificant.

The ACNGS turbine generator has been arranged in a peninsula orientation. With the exception of the Radwaste Building, this configuration excludes all major systems important to safety from the low trajectory turbine missile strike zones. The Radwaste Building does not contain any essential systems required for safe shutdown and is located below the turbine operating deck level. The location of the Radwaste System components relative to the turbine is such that they are adequately protected by the presence of the reinforced concrete pedestal, internal building walls and the turbine operating deck floor. Thus, the plant configuration complies with the guidelines of Regulatory Guide 1.115, "Protection Against Low Trajectory Turbine Missiles."

58

In addition to the above, due to the redundancy and testing features of the turbine overspeed protection, quality control manufacturing processes, materials, and inspection program, the hypothetical turbine missiles are considered very remote. Consequently the risk of potential turbine missile damage to safety related plant structures, systems, and components for the facility is acceptably low.



## SECTION 3.5 REFERENCES (Cont'd)

3.5-15	DELETED	58
3.5-16	DELETED	
3.5-17	DELETED	
3.5-18	D. R. Miller and W. A. Williams, "Tornado Protection for the Spent Fuel Storage Pool" APED 5696 of General Electric, November, 1968.	
3.5-19	W. F. Houghes and J. A. Brighton, "Theory and Problems of Fluid Dynamics," Schaum's Outline Series, Schaum Publishing Co., 1967.	21
3.5-20	M. A. Suarez, "Missile Generation by Fluid Propulsion," presented at ASCE, New York Metropolitan Chapter, March 26, 1974.	
3.5-21	Electric Power Research Institute, "Full-Scale Tornado-Missile Impact Tests," NP-148, Interim Report, April, 1976.	35(D)
3.5-22	U. S. Nuclear Regulatory Commission Standard Review Plan Section 3.5.1.3.	58

TABLE 3.5-2 HAS BEEN INTENTIONALLY DELETED

ACNGS-PSAR  
TABLE 3.5-3

TURBINE MISSILE CHARACTERISTICS  
(TYPICAL, FOR 38" WHEEL)

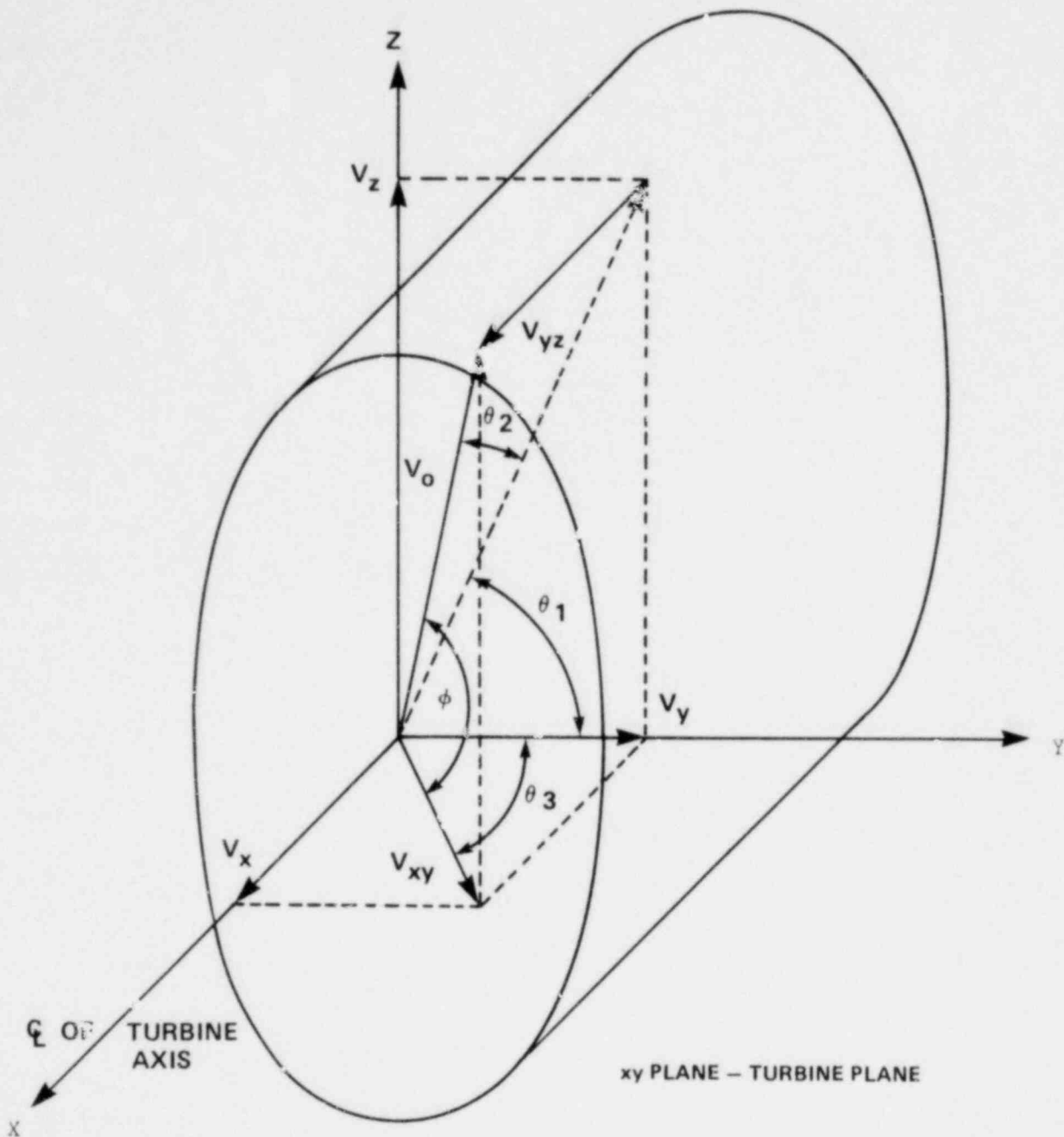
Fragment Angle, Deg.	120
Fragment Weight, Lb.	5944
Radius of CG, ft.	2.093
Polar Inertia, lb-ft-sec <sup>2</sup>	462
Min. Proj. Area, ft. <sup>2</sup>	3.657
Max. Proj. Area, ft. <sup>2</sup>	8.368
Failure Speed, Percent of 1800 RPM	169
Initial Velocity, ft/sec.	666.8
Energies, Million Ft-lb.	
Initial, Translation	41.0
Initial, Rotation	23.5
Outside Turbine Casing, Maximum Translation	20.5
After Air Drag, (Vertical Trajectory), Maximum	16.3

TABLE 3.5-4

PROBABILITY OF A TURBINE MISSILE IMPACTING PLANT  
SAFETY RELATED STRUCTURES

Structure	High Trajectory Missile Impact Probability (1)	Total Damage (2) Probability (yr <sup>-1</sup> )
Reactor Building	$2.5 \times 10^{-4}$	$1.0 \times 10^{-8}$
Control Building	$3.5 \times 10^{-4}$	$1.4 \times 10^{-8}$
Fuel Handling Building	$4.3 \times 10^{-4}$	$1.7 \times 10^{-8}$
DG Building	$1.7 \times 10^{-4}$	$6.9 \times 10^{-9}$
Reactor Auxiliary Building	$5.3 \times 10^{-4}$	$2.1 \times 10^{-8}$
UHS Intake Structure	$1.6 \times 10^{-4}$	$6.4 \times 10^{-9}$
		$7.5 \times 10^{-8}$

1. Based on missile deflection angle ( $\theta_2$ ) of + 25 and air resistance is neglected.
2. Based on missile generation probability of  $4.0 \times 10^{-5}$  per unit per year.



- $V_0$  : INITIAL VELOCITY
- $\theta_1$  : ANGLE FROM  $y$  AXIS TO  $V_{yz}$
- $\theta_2$  : ANGLE FROM  $yz$  PLANE TO  $V_0$
- $\theta_3$  : ANGLE ON THE GROUND
- $\phi$  : ANGLE FROM GROUND TO  $V_0$

AM.NO. 58, (5/81)

HOUSTON LIGHTING & POWER COMPANY  
Allens Creek Nuclear Generating Station  
Unit 1

ILLUSTRATION OF VARIABLES USED IN  
THE TURBINE MISSILE ANALYSIS

FIGURE 3.5-2

ACNGS-PSAR

LIST OF EFFECTIVE PAGES

CHAPTER 6  
ENGINEERED SAFETY FEATURES

<u>Page</u>	<u>Amendment No.</u>
1*	58
2*	58
3*	58
4*	58
5*	58
6*	58
6a	58
7*	58
8*	58
9*	58
10*	58
10a	58
11*	58
12*	58
13*	58
14*	58
i	58
ii	37
iii	46
iv	37
v	56
vi	56
vii	37
viii	37
ix	58
x	56
xi	37
xii	37
xiii	58
xifia	58
xiv	37
xv	56
xvi	56
xvii	56
xviii	56
xviiia	56
6.1-1	16
6.1-2	37
6.2-1	37
6.2-2	37
6.2-3	56

\* Effective Pages/Figures Listings

## ACNGS-PSAR

## LIST OF EFFECTIVE PAGES (Cont'd)

## CHAPTER 6

<u>Page</u>	<u>Amendment No.</u>
6.2-4	7
6.2-5	56
6.2-6	23
6.2-7	23
6.2-7a	17
6.2-8	1
6.2-9	40
6.2-10	39
6.2-11	23
6.2-12	37
6.2-13	23
6.2-14	23
6.2-14a	23
6.2-14b	23
6.2-15	6
6.2-16	37
6.2-17	37
6.2-18	23
6.2-18a	26
6.2-18b	37
6.2-19	37
6.2-19a	37
6.2-19b	17
6.2-19c	37
6.2-19d	17
6.2-20	23
6.2-21	26
6.2-22	37
6.2-23	26
6.2-24	6
6.2-25	6
6.2-26	37
6.2-27	23
6.2-28	26
6.2-29	6
6.2-29a	26
6.2-29a(i)	26
6.2-29b	37
6.2-29c	23
6.2-29c(i)	37
6.2-29d	6
6.2-29e	37
6.2-29f	37

## ACNGS-PSAR

## LIST OF EFFECTIVE PAGES (Cont'd)

## CHAPTER 6

<u>Page</u>	<u>Amendment No.</u>
6.2-29f(i)	37
6.2-29g	37
6.2-29h	37
6.2-29h(i)	37
6.2-29i	26
6.2-29j	26
6.2-29k	6
6.2-29l	37
6.2-29l(i)	37
6.2-29m	26
6.2-30	26
6.2-30a	23
6.2-30b	21
6.2-30c	23
6.2-30d	23
6.2-31	7
6.2-31a	7
6.2-32	57
6.2-32a	57
6.2-33	-
6.2-34	30
6.2-35	37
6.2-35a	30
6.2-35b	31
6.2-36	-
6.2-37	7
6.2-38	-
6.2-39	-
6.2-40	46
6.2-40a	21
6.2-41	26
6.2-41a	26
6.2-42	37
6.2-42a	16
6.2-42b	37
6.2-42c	37
6.2-43	37
6.2-43a	37
6.2-44	46
6.2-44a	37
6.2-45	37
6.2-46	37
6.2-46a	37



## ACNGS-PSAR

## LIST OF EFFECTIVE PAGES (Cont'd)

## CHAPTER 6

<u>Page</u>	<u>Amendment No.</u>
6.2-47	46
6.2-48	46
6.2-48a	46
6.2-49	37
6.2-49a	37
6.2-50	37
6.2-51	40
6.2-51a	10
6.2-52	-
6.2-53	-
6.2-54	37
6.2-55	46
6.2-55a	37
6.2-56	-
6.2-57	-
6.2-58	-
6.2-59	46
6.2-59a	46
6.2-60	57
6.2-61	57
6.2-61a	57
6.2-61b	57
6.2-62	57
6.2-63	-
6.2-64	57
6.2-65	17
6.2-66	-
6.2-67	23
6.2-68	23
6.2-68a	23
6.2-69	-
6.2-70	-
6.2-70a	23
6.2-71	57
6.2-71a	57
6.2-71b	8
6.2-72	28
6.2-72a	8
6.2-73	53
6.2-73a	53
6.2-74	37
6.2-75	57
6.2-75a	57
6.2-76	46

ACNGS-PSAR

LIST OF EFFECTIVE PAGES (Cont'd)

CHAPTER 6

<u>Page</u>	<u>Amendment No.</u>
6.2-77	53
6.2-77a	37
6.2-78	53
6.2-78a	37
6.2-78b	37
6.2-78c	37
6.2-78d	37
6.2-78e	39
6.2-78f	37
6.2-78f(i)	39
6.2-78g	39
6.2-78h	39
6.2-79	39
6.2-80	37
6.2-80a	23
6.2-80b	39
6.2-80b(i)	31
6.2-80c	31
6.2-80d	26
6.2-80e	26
6.2-80f	26
6.2-80g	26
6.2-81	5
6.2-81a	5
6.2-81b	21
6.2-82	23
6.2-83	26
6.2-84	28
6.2-85	28
6.2-86	23
6.2-87	37
6.2-88	23
6.2-89	37
6.2-90	23
6.2-91	26
6.2-91a	17
6.2-91b	5
6.2-91c	23
6.2-91d	23
6.2-92	17
6.2-93	17
6.2-94	37

ACNGS-PSAR

LIST OF EFFECTIVE PAGES (Cont'd)

CHAPTER 6

<u>Page</u>	<u>Amendment No.</u>
6.2-95	5
6.2-96	37
6.2-97	37
6.2-98	37
6.2-99	37
6.2-100	37
6.2-101	37
6.2-102	37
6.2-103	57
6.2-104	57
6.2-105	57
6.2-106	57
6.2-107	57
6.2-107a	57
6.2-107b	57
6.2-107c	57
6.2-107d	57
6.2-107e	57
6.2-107f	57
6.2-107g	57
6.2-107h	57
6.2-108	57
6.2-109	57
6.2-109a	57
6.2-109b	57
6.2-109c	57
6.2-109d	57
6.2-110 (Intentionally deleted)	57
6.2-111	58
6.2-112	58
6.2-113 (deleted)	37
6.2-114	23
6.2-115	17
6.2-116	37
6.2-117 (deleted)	37
6.2-118	5
6.2-119	5
6.2-119a	5
6.2-120	37
6.2-121	5
6.2-122	23
6.2-123	23
6.2-124	37

ACNGS-PSAR

LIST OF EFFECTIVE PAGES (Cont'd)

CHAPTER 6

<u>Page</u>	<u>Amendment No.</u>
6.2-125	37
6.2-126	37
6.2-127	37
6.3-1	56
6.3-1a	56
6.3-2	0
6.3-2a	0

ACNGS-PSAR

LIST OF EFFECTIVE PAGES (Cont'd)

CHAPTER 6

<u>Page</u>	<u>Amendment No.</u>
6.3-3	56
6.3-4	31
6.3-5	31
6.3-6	42
6.3-6a	42
6.3-7	42
6.3-7a	42
6.3-8	39
6.3-8a	37
6.3-8b	37
6.3-8c	56
6.3-8d	56
6.3-9	37
6.3-9a	5
6.3-10	31
6.3-10a	31
6.3-11	56
6.3-12	56
6.3-12a	56
6.3-12b	56
6.3-13	31
6.3-13a	31
6.3-14	5
6.3-14a	56
6.3-14b	37
6.3-14b(i)	37
6.3-14c	45
6.3-15	1
6.3-16	46
6.3-17	56
6.3-18	56
6.3-19	56
6.3-20	56
6.3-21	56
6.3-22	56
6.3-23	56
6.3-24	56
6.3-25	56
6.3-26	56
6.3-27	56
6.3-28	31
6.3-29	56
6.3-29a	3

## ACNGS-PSAR

## LIST OF EFFECTIVE PAGES (Cont'd)

## CHAPTER 6

<u>Page</u>	<u>Amendment No.</u>
6.3-30	56
6.3-31	-
6.3-31a	56
6.3-32	56
6.3-33	56
6.3-34	56
6.3-35	56
6.3-36	56
6.3-37	56
6.3-38	56
6.3-39	56
6.4-1	37
6.4-2	37
6.4-2a	32
6.4-2b	37
6.4-3	37
6.4-3a	37
6.4-4	37
6.4-5	37
6.4-5a	32
6.4-6	37
6.4-7	1
6.4-8	37
6.4-9	32
6.5-1	37
6.5-1a	1
6.5-2	42
6.5-2a	46
6.5-3	46
6.5-4	37
6.5-5	37
6.5-6	37
6.5-7	37
6.5-7a	37
6.5-8	-
6.6A-1	5
6.6A-2	5
6.6A-3	5
6.6A-4	5
6.6A-5	5

ACNGS-PSAR

LIST OF EFFECTIVE PAGES (Cont'd)

CHAPTER 6

<u>Page</u>	<u>Amendment No.</u>
6.6A-6	5
6.6A-7	5
6.6A-8	5
6.6A-9	5
6.6A-10	5
6.6A-11	5
6.6A-12	5
6.6A-13	5

## ACNGS-PSAR

## EFFECTIVE FIGURES LISTING\*

## CHAPTER 6

\*All figures, whether labelled "Unit 1" or "Units 1 and 2", are to be considered applicable to Unit No. 1.

<u>Figure No.</u>	<u>Amendment No.</u>
6.2-1	-
6.2-2	-
6.2-3	23
6.2-4	23
6.2-5	23
6.2-6	23
6.2-7	23
6.2-8	23
6.2-9	23
6.2-10	26
6.2-11	26
6.2-12	26
6.2-12 (Notes to Figure - 1 page)	26
6.2-13	26
6.2-14	-
6.2-15 (deleted)	5
6.2-16 (deleted)	5
6.2-17 (deleted)	5
6.2-18	37
6.2-19	23
6.2-20	37
6.2-21 (deleted)	37
6.2-22	17
6.2-23	17
6.2-24	17
6.2-25	46
6.2-26 (Sheet 1)	57
6.2-26 (Sheet 2)	57
6.2-26 (Sheet 3)	57
6.2-26 (Sheet 4)	57
6.2-26 (Sheet 5)	57
6.2-26 (Sheet 6)	57
6.2-26 (Sheet 7)	57
6.2-26 (Sheet 8)	57
6.2-26 (Sheet 9)	57
6.2-26 (Sheet 10)	57
6.2-26 (Sheet 11)	57



## EFFECTIVE FIGURES LISTING (Cont'd)

## CHAPTER 6

<u>Figure No.</u>	<u>Amendment No.</u>
6.2-26 (Sheet 12)	57
6.2-26 (Sheet 13)	57
6.2-26 (Sheet 14)	57
6.2-26 (Sheet 15)	57
6.2-26 (Sheet 16)	57
6.2-26 (Sheet 17)	57
6.2-26 (Sheet 18)	57
6.2-26 (Sheet 19)	57
6.2-26 (Sheet 20)	57
6.2-26 (Sheet 21)	57
6.2-26 (Sheet 22)	57
6.2-26 (Sheet 23)	57
6.2-26 (Sheet 24)	57
6.2-26 (Sheet 25)	57
6.2-26 (Sheet 26)	57
6.2-26 (Sheet 27)	57
6.2-26 (Sheet 28)	57
6.2-26 (Sheet 29)	57
6.2-26 (Sheet 30)	57
6.2-26 (Sheet 31)	57

## ACNGS-PSAR

## EFFECTIVE FIGURES LISTING (Cont'd)

## CHAPTER 6

<u>Figure No.</u>	<u>Amendment No.</u>
6.2-27a (deleted)	58
6.2-27b (deleted)	58
6.2-28	-
6.2-28a	53
6.2-29	53
6.2-30 (deleted)	37
6.2-31 (deleted)	37
6.2-32 (deleted)	37
6.2-33	23
6.2-34	23
6.2-35	37
6.2-36	5
6.2-37	5
6.2-38	5
6.2-39	5
6.2-40	5
6.2-41	5
6.2-42	5
6.2-43	5
6.2-44	37
6.2-45	5
6.2-46	23
6.2-47	37
6.2-48	5
6.2-49	23
6.2-50	23
6.2-51	23
6.2-52	37
6.2-53	5
6.2-54	5
6.2-55	5
6.2-56	5
6.2-57	5
6.2-58	5
6.2-59	5
6.2-60	8
6.2-61	8
6.2-62	21
6.2-63	37
6.2-64 (deleted)	37
6.2-65	37

## ACNGS-PSAR

## EFFECTIVE FIGURES LISTING (Cont:'d)

## CHAPTER 6

<u>Figure No.</u>	<u>Amendment No.</u>
6.2-66	23
6.2-67	30
6.2-68	30
6.3-1a	42
6.3-1b	42
6.3-2	42
6.3-3	56
6.3-4	-
6.3-5	-
6.3-5a	42
6.3-6	56
6.3-7	56
6.3-8a	-
6.3-8b	-
6.3-8c	-
6.3-9	56
6.3-10	56
6.3-11	56
6.3-12	56
6.3-13	56
6.3-13a	5
6.3-13b	5
6.3-13c	5
6.3-13d	5
6.3-14	56
6.3-14a	37
6.3-14b	5
6.3-15	56
6.3-15a (deleted)	46
6.3-15b (deleted)	46
6.3-16	56
6.3-17	56
6.3-18	56
6.3-19	56
6.3-20	56
6.3-21	56
6.3-22	56
6.3-23	56
6.3-24	56
6.3-25	56
6.3-26	56

## EFFECTIVE FIGURES LISTING (Cont'd)

## CHAPTER 6

<u>Figure No.</u>	<u>Amendment No.</u>
6.3-27	56
6.3-28	56
6.3-29	56
6.3-30	56
6.3-31	56
6.3-32	56
6.3-33	56
6.3-34	56
6.3-35	56
6.3-36	56
6.3-37	56
6.3-38	56
6.3-39	56
6.3-40	56
6.3-41	56
6.3-42	56
6.3-43	56
6.3-44	56
6.3-45	56
6.3-45a (deleted)	56
6.3-45b (deleted)	56
6.3-45c (deleted)	56
6.3-46	56
6.3-47	56
6.3-48	56
6.3-49	56
6.3-50	56
6.3-51	56
6.3-52	56
6.3-53	56
6.3-54	56
6.3-55	56
6.3-56	56
6.3-57	56
6.3-58	56
6.3-59	56
6.3-60	56
6.3-61	56
6.3-62	56
6.3-63	56
6.3-64	56
6.3-65	56
6.3-66	56
6.3-67	56
6.3-68	56

ACNGS-PSAR

EFFECTIVE FIGURES LISTING (Cont'd)

CHAPTER 6

<u>Figure No.</u>	<u>Amendment No.</u>
6.3-69	56
6.3-70	56
6.3-71	56
6.3-72	56
6.3-73	56
6.3-74	56
6.3-75	56
6A-1	5
6A-2	5
6A-3	5
6A-4	5
6A-5	51
6A-6	5

TABLE OF CONTENTS

CHAPTER 6

ENGINEERED SAFETY FEATURES

<u>Section</u>	<u>Title</u>	<u>Page</u>
6.1	<u>GENERAL</u>	6.1-1
6.2	<u>CONTAINMENT SYSTEMS</u>	6.2-1
6.2.1	CONTAINMENT FUNCTIONAL DESIGN	6.2-1
6.2.1.1	<u>Design Bases</u>	6.2-1
6.2.1.2	<u>System Design</u>	6.2-2
6.2.1.2.1	General	6.2-2
6.2.1.2.2	Shield Building	6.2-4
6.2.1.2.3	Containment Vessel	6.2-5
6.2.1.2.4	Drywell and Components	6.2-6
6.2.1.2.5	Pedestal and Reactor Shield Wall	6.2-8
6.2.1.2.6	Weir Wall and Horizontal Vents	6.2-9
6.2.1.2.7	Upper Pool	6.2-10
6.2.1.2.8	Pressure Suppression Pool	6.2-11
6.2.1.2.9	Containment Floors, Platforms and Rooms	6.2-13
6.2.1.3	<u>Design Evaluation</u>	6.2-15
6.2.1.3.1	Containment Transient Analysis	6.2-15
6.2.1.3.2	Subcompartment Transient Analysis	6.2-30
6.2.1.3.3	Shield Building Transient Analysis	6.2-31
6.2.1.3.4	Post-Accident Containment Pressure Calculation	6.2-32
6.2.1.3.5	Containment Pressure Calculation Inadvertent Actuation	6.2-32
6.2.1.4	<u>Testing and Inspection</u>	6.2-32
6.2.1.4.1	Provisions for Testing	6.2-32
6.2.1.4.2	Pre-Operational Leak Rate Testing	6.2-33
6.2.1.4.3	Initial Integrated Leak Rate Test	6.2-35a
6.2.1.4.4	Post-Operational Leak Rate Tests	6.2-37

LIST OF TABLES

<u>Table</u>	<u>Title</u>	<u>Page</u>
6.2-1a	Design Criteria for Containment System	6.2-82
6.2-1b	Supplementary Information for Water Pool Pressure Suppression Containment	6.2-86
6.2-2	Accident Chronology Steamline Break	6.2-89
6.2-3	Primary System Energy Distribution At the Time a Steam Line Break Occurs	6.2-90
6.2-4	Containment Subcompartment Analysis	6.2-91
6.2-5	Description of Assumptions Used in Shield Building Annulus Transient Analysis	6.2-92
6.2-6	Containment Response to a Loss-of-Coolant Accident	6.2-94
6.2-7	Open	6.2-95
6.2-8	Design Data for Standby Gas Treatment System Components	6.2-96
6.2-9	Properties of Filter Media-Activated Coconut Shell Charcoal	6.2-99
6.2-10	Single Failure Analysis - Standby Gas Treatment System	6.2-100
6.2-11	Summary of Tests	6.2-102
6.2-12	Containment Penetration and Isolation Valve Information	6.2-103
6.2-13	Intentionally Deleted	6.2-110
6.2-14	Comparison of Safety Related Air Filtration Systems with Regulatory Position of Regulatory Guide 1.52	6.2-111
6.2-15	Summary of Containment Analysis Results	6.2-114
6.2-16	Deleted	6.2-115
6.2-17	Energy Sources and Sinks for DBA Recirc and Main Steam Line Break Accident	6.2-116
6.2-18	Approximate Vent Flow Parameters	6.2-118
6.2-19	Loss of Coefficients Used in Vent Flow Model	6.2-119
6.2-20	Equivalent Loss Coefficients	6.2-119a
6.2-21	RWCU System Double Ended Line Break	6.2-120

## LIST OF FIGURES

<u>Figure</u>	<u>Title</u>
6.2-23	Shield Building Annulus Transient Analysis Pressure vs. Time (Lineal Scale)
6.2-24	Shield Building Annulus Transient Analysis Temperature vs. Time
6.2-25	Standby Gas Treatment System
6.2-26	Containment Isolation Valve Arrangements - Sheet 1
6.2-26	Containment Isolation Valve Arrangements - Sheet 2
6.2-26	Containment Isolation Valve Arrangements - Sheet 3
6.2-26	Containment Isolation Valve Arrangements - Sheet 4
6.2-26	Containment Isolation Valve Arrangements - Sheet 5
6.2-26	Containment Isolation Valve Arrangements - Sheet 6
6.2-26	Containment Isolation Valve Arrangements - Sheet 7
6.2-26	Containment Isolation Valve Arrangements - Sheet 8
6.2-26	Containment Isolation Valve Arrangements - Sheet 9
6.2-26	Containment Isolation Valve Arrangements - Sheet 10
6.2-26	Containment Isolation Valve Arrangements - Sheet 11
6.2-26	Containment Isolation Valve Arrangements - Sheet 12
6.2-26	Containment Isolation Valve Arrangements - Sheet 13
6.2-26	Containment Isolation Valve Arrangements - Sheet 14
6.2-26	Containment Isolation Valve Arrangements - Sheet 15
6.2-26	Containment Isolation Valve Arrangements - Sheet 16
6.2-26	Containment Isolation Valve Arrangements - Sheet 17
6.2-26	Containment Isolation Valve Arrangements - Sheet 18
6.2-26	Containment Isolation Valve Arrangements - Sheet 19
6.2-26	Containment Isolation Valve Arrangements - Sheet 20
6.2-26	Containment Isolation Valve Arrangements - Sheet 21
6.2-26	Containment Isolation Valve Arrangements - Sheet 22
6.2-26	Containment Isolation Valve Arrangements - Sheet 23
6.2-26	Containment Isolation Valve Arrangements - Sheet 24



## LIST OF FIGURES

<u>Figure</u>	<u>Titles</u>
6.2-26	Containment Isolation Valve Arrangements - Sheet 25
6.2-26	Containment Isolation Valve Arrangements - Sheet 26
6.2-26	Containment Isolation Valve Arrangements - Sheet 27
6.2-26	Containment Isolation Valve Arrangements - Sheet 28
6.2-26	Containment Isolation Valve Arrangements - Sheet 29
6.2-26	Containment Isolation Valve Arrangements - Sheet 30
6.2-26	Containment Isolation Valve Arrangements - Sheet 31
6.2-27a	Intentionally Deleted
6.2-27b	Intentionally Deleted
6.2-28	Time Required to Reach 4 Percent by Volume Hydrogen in Drywell vs. Peak Cladding Temperature - Conservative Temperature Distribution (Reference 6)
6.2-28a	Hydrogen Concentration in the Drywell (Without Blower Activation) -0.00023" Metal-Water Reaction
6.2-29	Hydrogen Concentration in the Drywell (Following Blower Activation)
6.2-30	Long Term Post Accident Containment and Drywell H <sub>2</sub> Concentration (Deleted)

ACNGS-PSAR

TABLE 6.2-14

COMPARISON OF SAFETY RELATED AIR FILTRATION SYSTEMS  
WITH REGULATORY POSITION OF REGULATORY GUIDE 1.52

Regulatory Position Item	Standby Gas Treatment System	ECCS Area Filtered Exhaust System	Control Room Emergency Filtration System
1a-e	All systems will comply with regulatory position.		System complies with regulatory positions except:
2a	System complies with regulatory position.	System complies with regulatory position	1) System does not include demisters. Within this system there is no source of entrained water droplets and therefore demisters are not required.  2) Electric heating coil is provided for humidity control.
2b	Physical separation will be provided between redundant components of all systems.		
2c	All systems will be designed as seismic Category I.		
2d	Not applicable. All systems are located outside Containment and therefore not subject to accident pressure surges.		
2e	All systems will comply with regulatory position and radiation levels given in Table 3.11-3.		
2f	All systems will comply with regulatory position. Flow rates are less than 30,000 cfm. HEPA filter arrangements will be limited to three high.		
2g	Pressure differential indicators will be provided locally for the demister, medium efficiency filter, pre-HEPA, after HEPA and across each ESF filtration train. Remotely located pressure differential indicators for the pre-HEPA and across each ESF filtration train will be provided in the Control Room. In addition, high limit alarm across the medium efficiency filter, pre-HEPA and across each ESF filtration train will be provided in the Control Room.		

## ACNGS-PSAR

TABLE 6.2-14 (Cont'd)

Regulatory Position Item	Standby Gas Treatment System	ECCS Area Filtered Exhaust System	Control Room Emergency Filtration System
	<p>Each ESF filtration train will be provided with a low flow indicator and low flow alarm in the Control Room. Temperature indicators will be provided locally at the inlet of the electric heating coil and the charcoal adsorber. These temperatures will be indicated in the Control Room. High charcoal adsorber bed temperature will be indicated and alarmed in the Control Room.</p>		
2h	Systems will comply with applicable IEEE standards (see Section 7.1).		
2i	Systems will be automatically activated upon the occurrence of a DBA by a redundant ESF signal.		
2j-1	All systems will comply with regulatory position.		
3a-j	All systems will comply with regulatory positions.		
3k	<p>System design includes provisions for preventing adsorber fires by ensuring continued cooling of adsorbers by air flow even in the event of a single failure. This is accomplished by providing cross connections between redundant fans and filter trains. In addition, system design includes provisions to apply water to the adsorber section, from the plant Fire Protection System, in the event of a fire in the adsorber beds.</p>		
3l-p	All systems will comply with regulatory position.		
4a-e	All systems will comply with regulatory position.		
5a-d	All systems will comply with regulatory positions.		
6a-b	All systems will comply with regulatory positions.		

FIGURES 6.2-27a & 6.2-27b HAVE BEEN  
INTENTIONALLY DELETED

ACNGS-PSAR

EFFECTIVE PAGE LIST  
CHAPTER 7

INSTRUMENTATION AND CONTROLS

PAGE NO.

AMENDMENT NO.

1*	58
1a*	58
2*	46
3*	44
4*	56
5*	40
6*	57
6a*	43
7*	44
8*	37
9*	57
10*	57
11*	57
11a	57
12*	40
13*	56
14*	37
15*	57
16*	37
17*	39
18*	37
19*	37
20*	39
21*	57
22*	37
23*	37
24*	37
i	44
ii	37
iii	37
iv	37
v	37
vi	37
vii	37
viii	40
ix	37
x	37
xi	37
xii	37
xiii	37
xiv	37
xv	37
xvi	37
xvii	37
xviii	37
xix	37
xx	37
xxi	58

ACNGS-PSAR  
EFFECTIVE PAGE LIST  
CHAPTER 7

INSTRUMENTATION AND CONTROLS

<u>PAGE NO.</u>	<u>AMENDMENT NO.</u>
xxii	37
xxiii	37
xxiv	37
xxv	37
xxvi	37
xxvii	37
xxviii	37
xxix	37
xxx	37
xxxi	37
xxxii	37
xxxiii	37
xxxiv	37
xxxv	37
xxxvi	37
xxxvii	37
xxxviii	37
xxxix	58
xl	37
xli	37
xlii	37
xliii	37
xliv	39
xlv	37
xlvi	37

TABLE OF CONTENTS

CHAPTER 7 (CONT'D)

<u>Section</u>	<u>Title</u>	<u>Page</u>
7.5.1.5.1	Loss of Habitability of Control Room	7.5-26
7.5.1.5.1.1	Criteria	7.5-26
7.5.1.5.1.2	Conditions Assumed to Exist as the Control Room Becomes Inaccessible	7.5-26
7.5.1.5.1.3	Description	7.5-27
7.5.1.5.1.4	Procedure for Reactor Shutdown from Outside Control Room	7.5-28
7.5.1.5.1.5	Controls and Instrumentation	7.5-29
7.5.1.5.1.5.1	Reactor Core Isolation Cooling (RCIC) System	7.5-29
7.5.1.5.1.5.2	Residual Heat Removal (RHR) System	7.5-30
7.5.1.5.1.5.3	Nuclear Boiler and Control Rod Drive System	7.5-31
7.5.1.5.1.5.4	Recirculation Flow Control System	7.5-33
7.5.1.5.1.5.5	Balance of Plant Systems	7.5-33
7.5.1.6	Safety Parameter Display System	7.5-32
7.5.2	ANALYSIS	7.5-33
7.5.2.1	<u>General</u>	7.5-33
7.5.2.2	<u>Normal Operation</u>	7.5-33
7.5.2.3	<u>Abnormal Transient Occurrences</u>	7.5-34
7.5.2.3.1	Shutdown and Isolation	7.5-34
7.5.2.4	<u>Accident Conditions</u>	7.5-34
7.5.2.4.1	Initial Accident Event	7.5-34
7.5.2.4.2	Post-Accident Tracking	7.5-34
7.5.2.4.2.1	Reactor Water Level and Pressure	7.5-34
7.5.2.4.2.2	Emergency Core Cooling	7.5-34
7.5.2.4.2.3	Containment and Reactor Vessel Isolation Control System	7.5-35
7.5.2.4.2.4	Standby Gas Treatment System	7.5-35
7.5.2.4.2.5	ECCS Area Filtered Exhaust System	7.5-37
7.5.2.4.2.6	Standby Power System	7.5-37

## ACNGS-PSAR

## LIST OF TABLES (CONT'D)

<u>Table</u>	<u>Title</u>	<u>Page</u>
7.3-17	ECCS Area Fan Coolers FMEA	7.3-187
7.3-18	Containment Vacuum Relief System FMEA	7.3-188
7.3-19	Control Room Alarms for Conditions that Render the Diesel Generator Unable to Respond to an Emergency Auto Start	7.3-190   43(U)
7.4-1	Reactor Core Isolation Cooling Instrument Specification	7.4-23
7.4-2	Reactor Shutdown Cooling Bypasses and Interlocks	7.4-24
7.5-0	Post-Accident Monitoring Instrumentation	7.5-44a   58
7.5-1	Containment and Reactor Vessel Isolation Control System Control Room I & C	7.5-45
7.5-2	Standby Gas Treatment System Control Room I & C	7.5-48
7.5-3	ECCS Area Filtered Exhaust System Control Room I & C	7.5-50
7.5-4	Standby Power System Control Room I & C	7.5-52
7.5-5	Essential Services Cooling Water System Control Room I & C	7.5-54
7.5-6	Control Room Air Conditioning Control Room I & C	7.5-56
7.5-7	ECCS Area Fan Coolers Control Room I & C	7.5-61
7.5-8	Panel Arrangement for Nuclenet 1000 Control Console	7.5-63
7.5-9	Specific Regulatory Design Requirements	7.5-69
7.5-10	Containment Vacuum Relief System	7.5-70
7.5-11	Nuclenet Control Panel Inserts	7.5-71
7.5-12	Standby Information Panel Inserts	7.5-72
7.5-13	Reactor Core Cooling Benchboard Inserts	7.5-73
7.6-1	Refueling Interlock Effectiveness	7.6-69
7.6-2	Process Radiation Monitoring Systems Characteristics	7.6-70
7.6-3	SRM System Trips	7.6-71
7.6-4	IRM Trips	7.6-72
7.6-5	LPRM System Trips	7.6-73
7.6-6	APRM System Trips	7.6-74
7.6-7	Reactor Water Cleanup Annunciators	7.6-75
7.7-1	Rod Control and Information System Instrument Specifications	7.7-39



ACNGS- PSAR

LIST OF EFFECTIVE PAGES  
CHAPTER 9  
AUXILIARY SYSTEMS

<u>Page No.</u>	<u>Amendment No.</u>
1*	58
2*	48
3*	58
4*	46
5*	47
6*	48
7*	46
8*	43
8a*	40
8b*	40
9*	58
10*	46
11*	40
i	53
iiii	37
iv	37
v	37
vi	46
vii	37
viii	37
ix	37
x	37
xi	46
xii	42
xiii	46
xiv	42
xv	40
xvi	37
xvii	53
xviii	37
xix	40
xx	40
xxi	40

\* Effective Pages/Figures Listings

ACNGS-PSAR  
EFFECTIVE PAGE LISTING  
CHAPTER 9  
AUXILIARY SYSTEMS

<u>PAGE NO.</u>	<u>AMENDMENT NO.</u>
9.2-11	37
9.2-11a	37
9.2-11b	37
9.2-12	37
9.2-13	37
9.2-13a	37
9.2-14	37
9.2-14a	37
9.2-15	58
9.2-15a	58
9.2-16	39
9.2-16a	37
9.2-16b	37
9.2-17c	37
9.2-17	37
9.2-18	58
9.2-18oa	58
9.2-18a & 19	37
9.2-20	37
9.2-21	37
9.2-21a	40
9.2-21b	37
9.2-22	37
9.2-23	37
9.2-23a	37
9.2-24	46
9.2-24a	46
9.2-25	46
9.2-26	46
9.2-27	46
9.2-28	46
9.2-29	37
9.2-29a	37
9.2-30	1
9.2-31	37
9.2-31a	22
9.2-32	37
9.2-33	37
9.2-34	37
9.2-35	37
9.2-36	22
9.2-37	22
9.2-38	22
9.2-39	37
9.2-40	-
9.2-41	37
9.2-42	46

ACNGS-PSAR

EFFECTIVE FIGURES LISTING

CHAPTER 9

AUXILIARY SYSTEMS

All figures, whether labelled "Unit 1" or "Units 1 and 2," are to be considered applicable to Unit No. 1

<u>Figure No.</u>	<u>Amendment No.</u>
9.1-1	-
9.1-2	-
9.1-3	37
9.1-3a	37
9.1-3b	42
9.1-4	-
9.1-5	-
9.1-6	-
9.1-7	-
9.1-8	-
9.1-9	-
9.1-10	39
9.1-11	39
9.1-12a	3
9.1-12b	3
9.1-12c	3
9.1-13	37
9.2-1	37
9.2-1a	37
9.2-1b	37
9.2-2	37
9.2-2a	37
9.2-2b	37
9.2-2c	37
9.2-3	37
9.2-4	37
9.2-5	37
9.2-6	37
9.2-7	37
9.2-8	37
9.2-8a	37
9.2-8b	37
9.2-8c	37
9.2-9 (deleted)	37
9.2-10	37
9.2-11	-
9.2-12	-
9.2-13	58
9.2-14	58

ACNGS- PSAR

The submerged evaporative pond is located to the south of the plant area with a diversion dike perpendicular to the 4,800 acre cooling lake shore. The submerged pond serves as the UHS. The normal source of essential services cooling water is the 4,800 acre cooling lake since water will be supplied from this heat sink whenever it is available. When using the 4,800 acre lake as the heat sink for essential services cooling the essential services, cooling water will be discharged from a seismic Category I structure located at the lake edge south of the UHS. The 50 acre submerged pond will perform its function as the Ultimate Heat Sink in the unlikely event of a loss of water from the 4,800 acre cooling lake. Switchover from the normal ESCWS mode of operation to the UHS mode of operation will be manual.

37(U)

A minimum level of 8 ft will be maintained in the 4,800 acre cooling lake at all times by creek inflows and by pumping from the Brazos River. In the event of a total loss of cooling water in the lake, the submerged pond will be more than adequate to permit emergency shutdown and cooldown for 4 months or in the event of an accident, to permit control of the accident for 4 months.

The intake is situated approximately 500 feet from the lake shore. The intake canal starts at the UHS bottom elevation 92 feet sloping downward continuously to the foot of the sill on UHS intake structure at elevation 86 feet. This corresponds to a 6 foot drop over a 139 foot horizontal run (approximately a 1:23 slope). The one foot sill will be provided as a barrier to limit the amount of silt which might enter UHS intake structure.

58

A causeway will provide access from the plant area to the intake structure. The crest of the UHS Causeway is established at EL 145.5 ft. The Causeway is designed for the Brazos River Probable Maximum Flood (PMF) at EL 135.4 ft, coupled with a wind set up of 0.7 ft and a wave run up of 8.2 ft generated by a 52 MPH Wind Velocity overwater (see PSAR Section 2.4 Appendix D, Table D4) This design flood consideration conservatively yields a design flood elevation of EL 144.2 ft, which will provide an adequate safe margin for the UHS Causeway.

37(U)

The Causeway slopes are protected against wave action by the placement of two (2) foot thick layers of soil-cement, measured perpendicular to the slope. The causeway slope stability analysis is described in Section 2.5 Appendix M Subsection M8. The design and placement of soil-cement will be in accordance with the ACNGS specifications, and subject to a strict construction quality control program.

58

9.2.5.3 Safety Evaluation

9.2.5.3.1 Consumptive Use

9.2.5.3.1.1 Methodology

In order to select an UHS design and to evaluate its expected performance, the following process has been used:

26

- I. Selection of Design Basis Accident (Section 9.2.5.3.1.2)

II.	Selection of Design Meteorological Conditions (Section 9.2.5.3.1.3)	26
	a. Maximum water temperature conditions	
	b. Maximum evaporation rate conditions	
III.	Simulation of the Design Basis Accident (Section 9.2.5.3.1.4)	37(C)
	a. Determination of the maximum Essential Services Cooling Water System (ESCWS) intake temperature.	
	b. Determination of the adequacy of a submerged 50 acre pond to provide four months supply of ESCWS water.	37(D)
9.2.5.3.1.2	Selection of Design Basis Accident	
	The Ultimate Heat Sink was evaluated for the following events:	26
	1. Safe shutdown	
	2. LOCA	37(U)
	These events were assumed to occur coincidentally with the loss of the cooling lake.	
	A comparison of these alternatives is presented in the following:	
	Table 9.2-5 indicates that the maximum instantaneous heat rejected to the Essential Services Cooling Water System for safe shutdown conditions without Fuel Pool Cooling is $286 \times 10^6$ Btu/hr. After 24 hours of cooldown operation, the total heat rejection would be reduced from $286 \times 10^6$ Btu/hr to approximately $100 \times 10^6$ Btu/hr.	37(D)
	Figure 9.2-11 presents the instantaneous heat rejection rate to the ESCWS as a function of time for safe shutdown.	37(U)
	Table 9.2-6 indicates that the maximum instantaneous heat rejected to the Essential Services Cooling Water System during and following a Design Basis LOCA is $230 \times 10^6$ Btu/hr without Fuel Pool Cooling.	37(D)

Under emergency shutdown conditions with the 4,800 acre lake available, the essential services cooling water discharge will be at the seismic Category I discharge structure located to the south of the UHS.

37(D)

Access to the UHS Intake Structure is via a manmade earth causeway extending from the main plant area and abutting the UHS Intake Structure. The causeway will be designed to be stable following a Safe Shutdown Earthquake or any other severe natural phenomena.

37(D)

58

Buried piping and conduit leading from the UHS Intake Structure, the UHS discharge structure and the ESCWS discharge structure will be designed to withstand the effects of the SSE and, in addition, will be designed to withstand a differential ground settlement of 3 inches per 1000 feet.

37(C)

An investigation was conducted to evaluate the potential sediment build up in the Ultimate Heat Sink. This supposition being that if an unusually large sediment deposit is postulated and is allowed to go uncorrected that it might adversely affect the cooling capacity of the UHS for plant shutdown.

The principal sources of sediment deposits are identified as Allens Creek and the makeup water for the Cooling Lake derived from Brazos River after prior passage through a 190 acre sedimentation basin. In order for sediment to enter the Essential Service Cooling Water System pumps it must pass over the one foot sill and then pass through trash bars and traveling water screens. The UHSIS has a flat floor at elevation 86 feet and the pumps are set back approximately 60 feet from the sill. The pump suction bells are 53 inches off the floor. This arrangement eliminates the possibility of degrading the pumps by the provision of sufficient clearance around pump suction areas.

Based upon the sediment deposit analysis and conservative engineering judgement, the sediments would most likely uniformly distribute over the entire cooling lake including the UHS. For such a uniform distribution, the study indicates that the total accumulated sediments in the UHS will most likely be less than one-half inch for 40 years of plant life. This is well within the 1 foot maximum deposition considered in the plant design.

58

A 1 foot sediment deposit will have no significant effect on the total inventory of required cooling water for an emergency cool down for 4 months, in the event of a total loss of the Cooling Lake. Since the total consumptive use for an emergency cooldown, including seepage and evaporation, will be in the order of 4 feet of a total of 8 feet water depth from a net storage of 404 acre-ft UHS.

Consideration was given to any slumping effect from the sediment deposit along the Heat Sink slopes. It has been recognized by the U.S. Army Corps of Engineers that for a slope having a 1 vertical to 20 horizontal or flatter, sediment will remain stable and no slumping or sliding effect will take place. Per PSAR Section 9.2.5.2 the intake canal will have 1:23 slope. As a



preventive measure, provisions will be made to flatten the upper 1 foot of the UHS side slopes and the UHS intake forebay canal bottom floor to 1 vertical and 20 horizontal or flatter.

In addition, in order to safeguard all the above, a monitoring program will be designed and implemented to check on any potential sediment accumulation in the UHS. The sediment monitoring program will be comprised of:

- (1) Establishing monument plates on the UHS floor. These plates will be located to provide representative indication of sediment deposition on flat and sloping surfaces in the Ultimate Heat Sink.
- (2) Probing will be conducted once per quarter for the first year of plant operation, and at least annually thereafter or until the monitoring program results indicate probing at a lesser frequency would not cause a safety hazard.
- (3) Probing operation will be performed in conjunction with the actual visual observation.
- (4) Sediment removal frequency will be determined as required according to the actual monitoring results during plant operation. However, the upper bound of the allowable limit of sediment build up will not exceed 1 foot on the bottom floor within the confinement of the Ultimate Heat Sink in the vicinity of the embankment and 6 inches in the intake canal proper.

In conclusion, based upon the sediment analysis results, the conservative considerations for the design and the sediment monitoring program, sediment deposition will not contribute any adverse effect on the functional capability of the Ultimate Heat Sink.

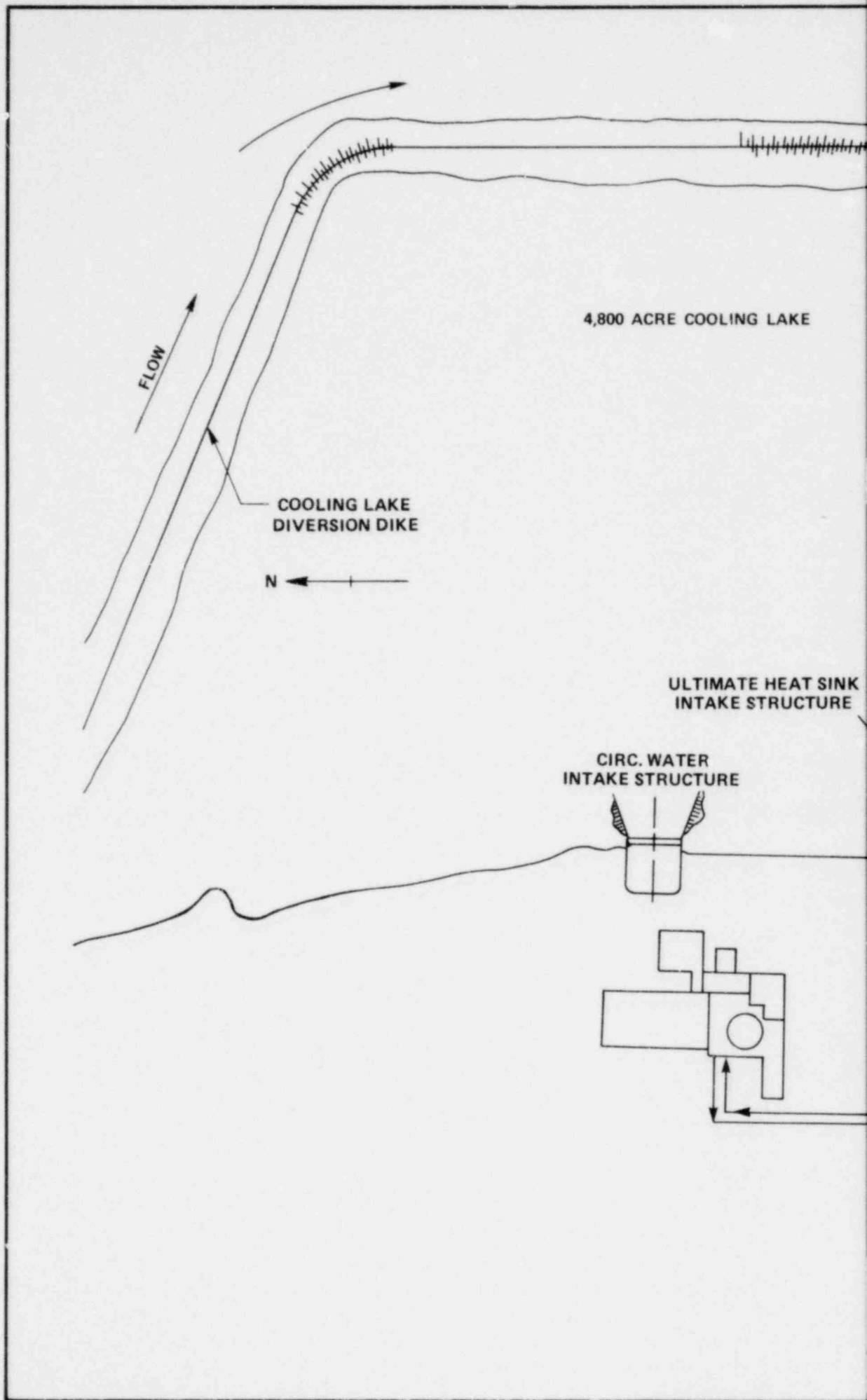
#### 9.2.5.3.3 Capability to Withstand a Single Failure

The UHS is designed to withstand the single failure of a manmade earthen structure or a single failure of any UHS active component.

In the event of failure of the cooling lake dam and complete loss of lake water, essential services cooling water will be drawn from the submerged pond.

58

37(D)



4,800 ACRE COOLING LAKE

FLOW

COOLING LAKE  
DIVERSION DIKE

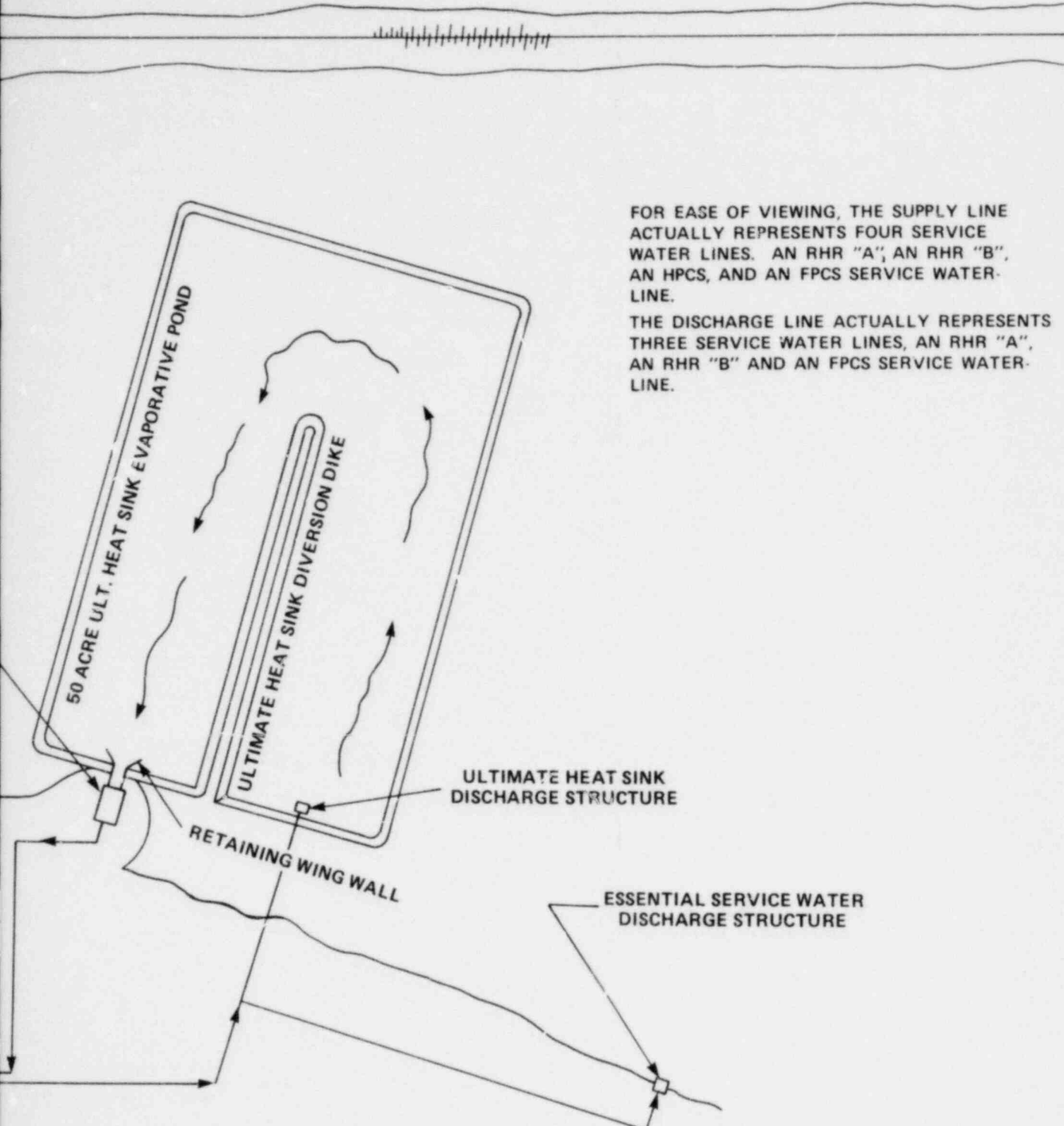
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ULTIMATE HEAT SINK  
INTAKE STRUCTURE

CIRC. WATER  
INTAKE STRUCTURE



FLOW



FOR EASE OF VIEWING, THE SUPPLY LINE ACTUALLY REPRESENTS FOUR SERVICE WATER LINES. AN RHR "A", AN RHR "B", AN HPCS, AND AN FPCS SERVICE WATER LINE.

THE DISCHARGE LINE ACTUALLY REPRESENTS THREE SERVICE WATER LINES, AN RHR "A", AN RHR "B" AND AN FPCS SERVICE WATER LINE.

ULTIMATE HEAT SINK DISCHARGE STRUCTURE

RETAINING WING WALL

ESSENTIAL SERVICE WATER DISCHARGE STRUCTURE

AM. NO. 58, (5/81)

HOUSTON LIGHTING & POWER COMPANY  
Allens Creek Nuclear Generating Station  
Unit 1

ULTIMATE HEAT SINK SCHEMATIC  
(SHEET 1)  
FIGURE 9.2-13

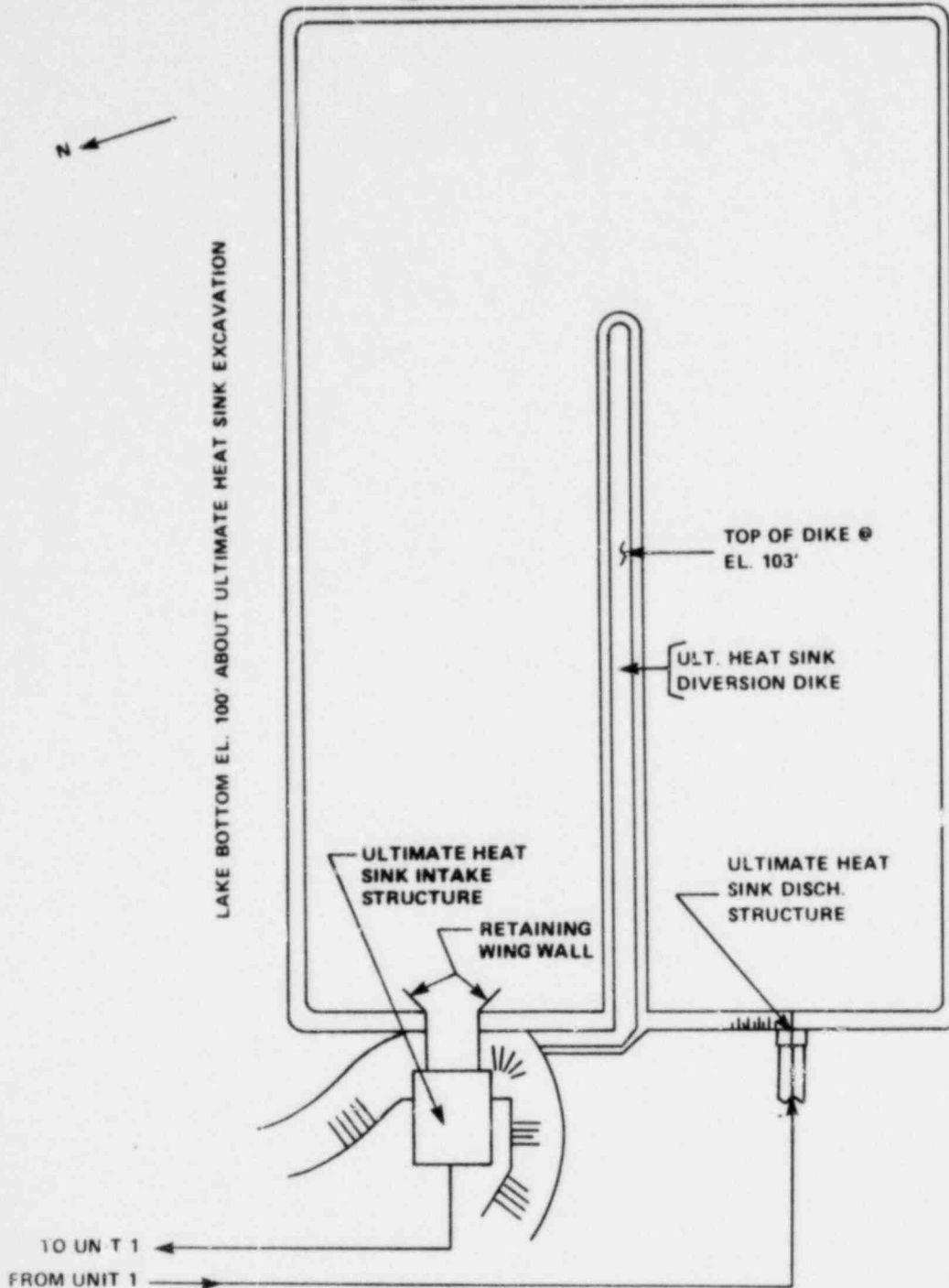
ACNGS - PSAR  
ULTIMATE HEAT SINK - EXCAVATION

4,800 ACRE COOLING LAKE

GRADE EL. 100' ± (TYP.)



LAKE BOTTOM EL. 100' ABOUT ULTIMATE HEAT SINK EXCAVATION



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HOUSTON LIGHTING & POWER COMPANY  
Allens Creek Nuclear Generating Station  
Unit 1

ULTIMATE HEAT SINK SCHEMATIC  
(SHEET 2)

FIGURE 9.2-14

ACNGS-PSAR  
LIST OF EFFECTIVE PAGES  
CHAPTER 12

RADIATION PROTECTION

<u>Page</u>	<u>Amendment No.</u>
1*	58
2*	58
2a*	58
3*	45
4*	37
5*	37
i	37
ii	46
iii	37
iiia	58
iv	37
v	37
vi	37
12.1-1	57
12.1-1a	57
12.1-1b	57
12.1-2	35
12.1-3	41
12.1-3a	41
12.1-3b	41
12.1-3c	41
12.1-4	41
12.1-5	35
12.1-6	39
12.1-7	37
12.1-8	35
12.1-8a	35
12.1-8b	35
12.1-9	35
12.1-10	-
12.1-11	39
12.1-12	35
12.1-12a	35
12.1-13	35
12.1-14	22
12.1-14a	35
12.1-15	35
12.1-16	35
12.1-17	35
12.1-18	35
12.1-19	41
12.1-19a	41
12.1-20	45
12.1-20a	45
12.1-21	45

\*Effective Pages/Figures Listings

ACNGS-PSAR  
LIST OF EFFECTIVE PAGES (Cont'd)  
CHAPTER 12

<u>Page</u>	<u>Amendment No.</u>
12.1-22	45
12.1-22a	45
12.1-22b	45
12.1-22c	45
12.1-23	45
12.1-23a	45
12.1-23b	45
12.1-24	45
12.1-25	35
12.1-26	35
12.1-26a	39
12.1-26b	39
12.1-26c	35
12.1-27	-
12.1-27a	35
12.1-27b	35
12.1-28	-
12.1-28a	35
12.1-28b	35
12.1-29	35
12.1-30	35
12.1-31	-
12.1-32	-
12.1-33	-
12.1-34	-
12.1-35	-
12.1-36	-
12.1-36a	35
12.1-36b	35
12.1-36c	35
12.1-36d	35
12.1-36e	35
12.1-36f	35
12.1-36g	35
12.1-36h	35
12.1-37	35
12.1-38	35
12.1-39	-
12.1-40 (deleted)	45
12.1-41	45
12.1-42	3
12.1-43	35
12.1-44	3
12.1-45	35
12.1-46	45
12.1-47	45
12.1-48	45
12.1-49	45

ACNGS-PSAR  
LIST OF EFFECTIVE PAGES  
CHAPTER 12

<u>Page</u>	<u>Amendment No.</u>
12.1-50	45
12.2-1	43
12.2-1a	43
12.2-1b	46
12.2-2	41
12.2-3	37
12.2-4	39
12.2-5	39
12.2-50a	39
12.2-5a	57
12.2-6	17
12.2-7	17
12.2-8	42
12.2-80a	42
12.2-8a	39
12.2-8a(i)	35
12.2-8b	35
12.2-8c	17
12.2-9	1
12.2-10	-
12.2-11	-
12.2-12	-
12.2-13	-
12.2-14	35
12.2-15	35
12.2-16	46
12.2-17	35
12.2-18	21
12.2-19	35
12.2-19a	57
12.2-20	35

## ACNGS-PSAR

## LIST OF TABLES

<u>Table</u>	<u>Title</u>	<u>Page</u>
12.2-2	Containment Inhalation Dose	12.2-16
12.2-3	Radwaste Building Inhalation Dose	12.2-17
12.2-4	Maximum Annual Inhalation Dose to Plant Personnel	12.2-18
12.2-5	Airborne Radiation Monitoring System Characteristics	12.2-19
12.2-5a	Accident Release Monitoring Points	12.2-19a
12.2-6	Steam and Liquid Leaks for Determining Airborne Activity and Ventilation Release	12.2-20
12.2-7	Noble Gas/Off-Gas Release Comparison of Calculational Model	12.2-21
12.2-8	Radionuclide Reactor Water Concentration Comparison of Calculational Model With 1971 GE Source Terms and Monticello	12.2-22
12.2-9	Pressure Transient Comparison of Calculational Model With Monticello Measurements	12.2-23
12.2-10	Suppression Pool Partitioning	12.2-24
12.2-11	Summary of Activity in the Containment Building With Time After Isolation Scram	12.2-25
12.2-12	Source Strength in Containment Building Ventilation Air With Time After Isolation Scram Summation of Gamma Energies	12.2-27

## ACNGS-PSAR

LIST OF EFFECTIVE PAGES  
CHAPTER 13  
CONDUCT OF OPERATIONS

<u>Page</u>	<u>Amendment</u>
1*	58
2*	57
3*	57
4*	57
5*	57
i	58
ii	55
iii	55
iiia	55
iiib	55
iv	58
v	57
13.1-1	57
13.1-2	57
13.1-3	57
13.1-4	57
13.1-5	57
13.1-6	57
13.1-7	57
13.1-8	57
13.1-9	57
13.1-10	57
13.1-11	57
13.1-12	57
13.1-13	57
13.1-14	57
13.1-15	57
13.1-16	33 (deleted)
13.1-16a	33 (deleted)
13.1-17	33 (deleted)
13.1-17a	33 (deleted)
13.1-18	33 (deleted)
13.1-19	33 (deleted)
13.1-19a	33 (deleted)
13.1-20	33 (deleted)
13.1-21	33 (deleted)
13.1-22	33 (deleted)
13.1-23	33 (deleted)
13.1-24	33 (deleted)
13.1-24a	33 (deleted)
13.1-25	33 (deleted)
13.1-26	33 (deleted)

\*Effective Pages/Figures Listing

## ACNGS-PSAR

## CHAPTER 13

CONDUCT OF OPERATIONS

<u>Section</u>	<u>Title</u>	<u>Page</u>	
13.0	<u>ORGANIZATIONAL STRUCTURE OF APPLICANT</u>	13.1-1	
13.1	<u>ORGANIZATIONAL STRUCTURE OF APPLICANT</u>	13.1-1	
13.1.1	MANAGEMENT AND TECHNICAL SUPPORT ORGANIZATION	13.1-1	
13.1.1.1	<u>Design and Operating Responsibilities</u>	13.1-1	
13.1.1.2	<u>Organizational Arrangements</u>	13.1-5	
13.1.1.3	<u>Qualifications</u>	13.1-8	33(U)
13.1.1.4	<u>Qualifications of Corporate Personnel</u>	13.1-8	
13.1.1.5	<u>Project Management Organization</u>	13.1-12	
13.1.2	OPERATING ORGANIZATION	13.1-43	
13.1.2.1	<u>Plant Organization</u>	13.1-43	
13.1.2.2	<u>Personnel Functions, Responsibilities and Authorities</u>	13.1-43	
13.1.2.3	<u>Shift Crew Composition</u>	13.1-48	
13.1.3	QUALIFICATION REQUIREMENTS FOR NUCLEAR PLANT PERSONNEL	13.1-48	
13.1.3.1	<u>Minimum Qualification Requirements</u>	13.1-48	
13.1.3.2	<u>Qualifications of Plant Personnel</u>	13.1-48	
APPENDIX 13.1A	<u>HL&amp;P ACNGS PERSONNEL RESUMES</u>	13.1A-1	33(U)
13.2	<u>TRAINING PROGRAM</u>	13.2-1	
13.2.1	PROGRAM DESCRIPTION	13.2-1	
13.2.1.1	<u>Program Content</u>	13.2-1	
13.2.1.2	<u>Coordination with Preoperational Tests and Fuel Loading</u>	13.2-3	
13.2.1.3	<u>Practical Reactor Operation</u>	13.2-3	33(U)
13.2.1.4	<u>Reactor Simulation Training</u>	13.2-3	
13.2.1.5	<u>Previous Nuclear Training</u>	13.2-4	



ACNGS-PSAR  
LIST OF TABLES

<u>Table</u>	<u>Title</u>	<u>Page</u>	
13.1-1	Minimum Qualification Requirements For Onsite Personnel	13.1-49	33(U)
13.1A-1	HL&P ACNGS Personnel Summary	13.1A-10	
13.1-2	Project Technical Qualifications and Resources	13.1-50	
13.1-3	Corporate Resources	13.1-51	58
13.2-1	Technical Specialist Training	13.2-8	33(U)
13.3-1	ACNGS Emergency Response Organization Drills and Exercises	13.3-35	55

ACNGS- PSAR  
LIST OF EFFECTIVE PAGES

CHAPTER 15  
ACCIDENT ANALYSIS

<u>Page</u>	<u>Amendment</u>
1*	58
1a*	56
2*	42
3*	56
3a	56
4*	56
5*	56
6*	57
7*	56
8*	57
9*	58
10*	35
i	35
ii	35
iii	41
iv	35
v	42
vi	35
vii	35
viii	41
15.1-1	56
15.1-1a	56
15.1-2	-
15.1-3	-
15.1-3a	26
15.1-4	5
15.1-5	35
15.1-6	56
15.1-7	42
15.1-8	35
15.1-9	35
15.1-10	35
15.1-11	35
15.1-11a	35
15.1-12	35
15.1-13	35
15.1-14	35
15.1-15	35
15.1-16	35
15.1-17	35
15.1-18	35
15.1-18a	35
15.1-19	35
15.1-20	35

\*Effective Pages/Figures Listings

ACNGS-PSAR

EFFECTIVE PAGES LISTING  
APPENDIX 15B

ALLENS CREEK NUCLEAR GENERATING STATION  
RELIABILITY ANALYSIS PROGRAM

<u>Page</u>	<u>Amendment No.</u>
15B-1	57
15B-2	57
15B-3	57
15B-4	57
15B-5	57

ACNGS-PASR

LIST OF EFFECTIVE PAGES  
APPENDIX C

<u>Page No.</u>	<u>Amendment No.</u>
1*	58
2*	58
3*	57
i	42
ii	42
iii	42
iv	58
v	42
vi	42
vii	42
viii	42
Cl.1-1	17
Cl.2-1	17
Cl.3-1	35
Cl.4-1	17
Cl.5-1	35
Cl.6-1	35
Cl.7-1	35
Cl.8-1	42
Cl.9-1	35
Cl.10-1	35
Cl.11-1	17
Cl.12-1	35
Cl.13-1	35
Cl.14-1	17
Cl.15-1	17
Cl.16-1	35
Cl.17-1	35
Cl.18-1	35
Cl.19-1	17
Cl.20-1	42
Cl.21-1	35
Cl.22-1	31
Cl.23-1	35
Cl.24-1	17
Cl.25-1	35
Cl.26-1	42
Cl.27-1	42
Cl.28-1	46
Cl.29-1	42
Cl.29-2	42
Cl.30-1	45
Cl.31-1	42
Cl.31-2 (deleted)	42
Cl.31-3 (deleted)	42

\* Effective Pages/Figures List

## ACNGS-PASR

LIST OF EFFECTIVE PAGES (Cont'd)  
APPENDIX C

<u>Page No.</u>	<u>Amendment No.</u>
Cl.32-1	35
Cl.33-1	42
Cl.34-1	17
Cl.35-1	35
Cl.36-1	22
Cl.37-1	45
Cl.38-1	46
Cl.38-2	42
Cl.39-1	46
Cl.40-1	17
Cl.41-1	17
Cl.42-1	35
Cl.43-1	46
Cl.44-1	56
Cl.45-1	17
Cl.46-1	22
Cl.47-1	35
Cl.48-1	35
Cl.49-1	35
Cl.50-1	35
Cl.50-2	35
Cl.51-1	35
Cl.52-1	58
Cl.53-1	35
Cl.54-1	46
Cl.55-1	17
Cl.56-1	18
Cl.57-1	22
Cl.58-1	46
Cl.59-1	42
Cl.60-1	42
Cl.60-2	17
Cl.61-1	22
Cl.62-1	22
Cl.63-1	22
Cl.63-2	22
Cl.64-1	46
Cl.65-1	35
Cl.66-1	42
Cl.67-1	22
Cl.68	45
Cl.68-2 (deleted)	45
Cl.68-3 (deleted)	45
Cl.68.1-1	42
Cl.68.2-1	42
Cl.69-1	44
Cl.69-2	44
Cl.69-3	44
Cl.69-4	44

## APPLICANT'S REGULATORY GUIDE POSITIONS

## TABLE OF CONTENTS (Cont'd)

<u>Number</u>	<u>Title</u>	
1.46	Protection Against Pipe Whip Inside Containment (Rev. 0, 5/73)	35(C)
1.47	Bypassed and Inoperable Status Indication for Nuclear Power Plant Safety Systems (Rev. 0, 5/73)	35(C)
1.48	Design Limits and Loading Combinations for Seismic Category I Fluid System Components (Rev. 0, 5/73)	35(C)
1.49	Power Levels of Water-Cooled Nuclear Power Plants (Rev 1, 12/73)	35(C)
1.50	Control of Preheat Temperature for Welding of Low-Alloy Steel (Rev. 0, 5/73)	35(C)
1.51	Inservice Inspection of ASME Code Class 2 and 3 Nuclear Power Plant Components (withdrawn)	35(C)
1.52	Design, Testing and Maintenance Criteria for Engineered-Safety - Feature Atmosphere Cleanup System Air Filtration and Adsorption Units of Light-Water-Cooled Nuclear Power Plants (Rev. 2, 3/78)	35(C)   58
1.53	Application of the Single-Failure Criterion to Nuclear Power Plant Protection Systems (Rev. 0, 6/73)	35(C)
1.54	Quality Assurance Requirements for Protection Coatings Applied to Water-Cooled Nuclear Power Plants (Rev. 0, 6/73)	5 
1.55	Concrete Placement in Category I Structures (Rev. 0, 6/73)	35(C)
1.56	Maintenance of Water Purity in Boiling Water Reactors (Rev. 0, 6/73)	
1.57	Design Limits and Loading Combinations for Metal Primary Reactor Containment System Components (Rev. 0, 6/73)	35(C)
1.58	Qualification of Nuclear Power Plant Inspection, Examination and Testing Personnel (Rev. 0, 8/73)	35(C)
1.59	Design Basis Floods for Nuclear Power Plants (Rev. 2, 8/77)	42(U)
1.60	Design Response Spectra for Seismic Design of Nuclear Power Plants (Revision 1, 12/73)	
1.61	Damping Values for Seismic Design of Nuclear Power Plants (Rev. 0, 10/73)	35(C)
1.62	Manual Initiation of Protective Actions (Rev. 0, 10/73)	35(C)

ACNGS-PSAR  
REGULATORY GUIDE 1.52  
(Rev. 2, 3/78)

58  
35(D)

DESIGN, TESTING AND MAINTENANCE CRITERIA FOR ENGINEERED-SAFETY-FEATURE  
ATMOSPHERE CLEANUP SYSTEM AIR FILTRATION AND ADSORPTION UNITS OF LIGHT-  
WATER-COOLED NUCLEAR POWER PLANTS

Applicant's Position:

A comparison of safety related air filtration systems with the requirements  
of this Regulatory Guide is given in Table 6.2-14 of the PSAR.