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CALCULATION NO. PSL-BESM-98-005 REV 0

SHEET NO. 1

1.0 Purpose/Scope

GL 92-08 (Ref. 2.1) has required FPL to review the ampacity correction factors (ACF) used for raceway with fire barriers. The ampacity correction factors were updated by calculation PTN-BFJM-96-005 and were based on testing performed at Omega Point Laboratories. The NRC in Reference 2.2 has expressed concern over the testing performed at Omega Point Laboratories; therefore, this calculation will determine applicable ampacity correction factors for St. Lucie based on testing performed at Underwriters Laboratories. This calculation will use heat transfer relationships to extrapolate the results from the tested fire barriers to thicknesses which bound the thickness of fire barrier used at the St. Lucie Plant Units 1 and 2. This calculation is intended to be a conservative extrapolation of test data based on the laws of heat transfer and not a thorough heat transfer evaluation.

2.0 References

- 2.1 GL-92-08, "Thermo-Lag 330-1 Fire Barriers" Dated December 17, 1992.
- 2.2 Second Request for Additional Information Generic Letter 92-08 "Thermo-Lag 330-1 Fire Barriers, St. Lucie Plant Units 1 and 2 and Turkey Point Plant Units 3 and 4", (TAC NO. M82809), Dated April 23,1998 Addressed to T.F. Plunkett and signed by Fredric J. Hebdon, Director
- 2.3 ASHRAE Handbook, 1991 Fundamentals
- 2.4 NRC Safety Evaluation Addressing Thermo-Lag Related Ampacity Derating Issues for Crystal River (TAC NO. M91772), Dated November 14, 1997, Addressed to Roy A. Anderson and Signed by L. Raghaven, Project Manager
- 2.5 ANSI C80.1-1990, Table 2 "Dimensions and Weights of Rigid Steel Conduits"
- 2.6 Underwriters Laboratories, Ampacity Test Investigation of Raceway Fire Barriers for Conduit and Cable Tray Systems, Dated May 8, 1996, File NC1973, Project 95NK17030 (Note: Recorded in Passport as REPORT NC1973)
- 2.7. TSI Inc., Thermo-Lag 330 & 770 Thermal Properties (Included as Attachment 1)
- 2.8. NEMA Publication WC3-1980, Rubber-Insulated Wire and Cable for the Transmission and Distribution of Electrical Energy.



Heat transfer will be calculated per foot of raceway length in accordance with the following relationship:

 $q = (T_c - T_a) / (R_i + R_g + R_c + R_g)$

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- q = Rate of heat transfer from raceway
- $T_c = Temperature of conductor (90°C/194°F)$
- $T_a = Ambient temperature (40°C/104°F)$
- R_i = Thermal resistance of all items within the raceway including the raceway itself
- $R_g =$ Thermal resistance of the air gap between the raceway and the fire barrier material
- R_t = Thermal resistance of the fire barrier material
- R_s = Thermal resistance at the surface of the protected or unprotected raceway

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The heat transferred from the raceway under steady state conditions is essentially equal to the I²R losses within the conductors. These heat values can be determined from the test data based on the measured current and size of conductor used.

T_c and T_a are fixed test parameters with values which are listed above.

The thermal resistance values will be determined based on test data and physical properties as follows:

R, will be calculated from the test data for raceway without fire barrier.

 R_g will be calculated from test data for raceway with a fire barrier of tested thickness.

 R_t will be calculated based on the known thermal conductivity (k) for Thermo-Lag material.

R_s will be based on known physical properties and the laws of convection and radiation heat transfer.

After all of the thermal resistance values have been established, the heat transferred can be calculated for the raceway with a desired thickness of fire barrier by recalculating R_c and R_s considering the additional thickness.

Since the heat is a function of the current squared, the ampacity correction factor (ACF) will be determined by the following relationship.

ACF = $I_p/I = (q_p/q)^{1/2}$ where the subscript p refers to the protected raceway

As a test of the methodology, the test data for 1 hour fire barrier will be used to predict the ACF for the 3 hour barrier test. These results will be compared to the test data to demonstrate the conservatism of the methodology.

4.0 Assumptions/Bases

- 4.1 The total heat load used in the extrapolation of the ampacity correction factors associated with fire barriers will be based on the I²R losses in the cables which will be representative of the total heat load. The testing documented in Reference 2.6 included paired sets of conductors with the same current running in opposite directions; therefore, the magnetic fields associated with this current will be effectively canceled. Generally, inductive losses are minimal in plant applications due to the practice of routing three phases of power cables in the same raceway. Inductive losses are accounted for in the ampacity rating calculations for the cables.
- 4.2 Surface emittance for cable, raceway, and Thermo-Lag will be assumed to be equal to 0.9. Note that a high emittance value will reduce the thermal resistance at the surface having an overall effect of maximizing the ampacity de-rating from the additional thickness of Thermo-Lag.



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- 4.3 Heat transfer through the sides of cable tray will be assumed to be zero. This will reduce the heat transfer equation for tray to a one dimensional heat transfer equation. As the tested cable tray is relatively wide ,24", compared to the depth, 4", this test is expected to be a good approximation for all cable tray widths.
 - 4.4 The thickness of the Thermo-Lag in the tests is assumed to be at the minimum allowable thickness specified. This thickness will provide a conservative ACF value as it maximizes the thickness of Thermo-Lag which must be added to reach the final thickness.

C	onduit	5			
1	Hour	Thermo-Lag	330-1	0.625 Inches	(Ref. 2.6 Page 6)
3	Hour	Thermo-Lag	330-1	1.25 Inches	•
\mathbf{T}_{2}	ray			<i>.</i> * ,	<u>n</u>
1	Hour	Thermo-Lag	330-1	0.625 Inches,	
3	Hour	Thermo-Lag	330-1	1.125 Inches	

- 4.5 The calculation will be performed assuming the following bounding plant configurations:
 - Conduit With 1 Hour Barrier Bounded by 1 to 4" Conduit Maximum Barrier Thickness = 1-1/2"
 - Conduit With 3 Hour Barrier Bounded by 1 to 4" Conduit Maximum Barrier Thickness = 3-1/16"
 - Tray or Banked Conduit With 1 Hour Barrier Bounded by 4" deep tray and 1 to 4" Banked Conduit Maximum Barrier Thickness = 1-1/2"
 - Tray or Banked Conduit With 3 Hour Barrier Bounded by 4" deep tray and 1 to 4" Banked Conduit Maximum Barrier Thickness = 3-1/16"

Adjacent layers of fire barriers are assumed to be installed with a layer of trowel grade material creating a homogeneous thickness of Thermo-Lag material with no intervening air gaps. An exception is the 1 hour upgrade which provides a second layer of Thermo-Lag 330 applied directly on the base layer. For this case, the potential for additional thermal resistance at this interface will be ignored. The conservative assumptions relative to the Thermo-Lag thickness applied will compensate for any additional resistance at this interface.

- 4.6 Raceway is made of rigid steel, magnetic material, which is typical for power plant installations.
- 4.7 Banked conduit which is banked in a single plane can be assumed to be equivalent to cable tray. Both configurations involve a cable mass arranged in a shallow rectangular section. Both configurations involve an air gap between the cables and the fire barrier material.



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- 4.8 The thermal resistance values for all items within the raceway and for the gap between the conduit and the Thermo-Lag material will be assumed to remain constant as additional thickness of Thermo-Lag is installed. Considering that the geometry of these areas is not changed, this approximation is reasonable.
- 4.9 This calculation is valid for indoor areas where the surrounding air and surface temperatures are relatively equal. Air flow around the raceway is assumed to be in the laminar flow region.
- 5.0 Calculation
- 5.1 Determination of test heat loads

Test heat loss in watts is calculated by the following equation:

q=I²RN

q= Heat Per Foot I= Test Current R= Cable Resistance Per Foot N= Number of Conductors

Raceway Size	(Conductor)	Test Current ⁽¹⁾	Resistance Per Foot ⁽²⁾	Number of Conductors	Heat/Ft Watts	Heat/Ft BTU/Hr ⁽³⁾
1″	(1-4C/#10)	30.5	.00136	4	5.06	17.27
.1" w/3 Hr	Barrier	31.8			5.50	18.78
4″	(12-3C/#6)	27.2	.000548	36	14.60	49.81
4" w/1 Hr	Barrier	28.1			15.58	53.17
4"	(12-3C/#6)	26.0	.000548	36	13.34	45.52
4" w/3 Hr	Barrier	25.3			12.63	43.10
Tray	(96-3C/#6)	28.8	.000548	288	130.91 .	446.78
Tray w/l	Hr Barrier	17.0			45.61	155.67
Tray	(96-3C/#6)	28.0	.000548	288	123.73	422.30
Tray w/3	Hr Barrier	16.4			42.45	144.87

1. Normalized test current is from Reference 2.6

2. Resistance per foot is from Ref. 2.8 Section 2.5, Table 2-6, Table 6-1

3. Multiply Watts by 3.413 to obtain BTU/Hr



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CALCULATION SHEET
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5.2 Determination of Thermo-Lag R values (R.)
      For heat transfer through Thermo-Lag cylinder
             R = Ln(Ro/Ri)/2\pi kL
                                                                    (Ref. 2.3, Page 2.3)
             Ro= Outside Radius
             Ri= Inside Radius
             k= Thermal Conductivity = 0.1 BTU/Hr-FT-°F (Ref. 2.7)
             L= Length = 1 Ft. (Per Foot)
      For heat transfer through Thermo-Lag sheet
             R= L/kA
                                                                    (Ref. 2.3, Page 2.3)
             L= Thickness
             k= Thermal Conductivity = 0.1 BTU/Hr-FT-°F (Ref. 2.7)
             A= Surface Area
      A full tabulation of the Thermo-Lag R values for the various sizes is included in the
      spreadsheet below.
5.3 Determination of surface R values (R_s)
      The surface resistance will consider free convection and radiation heat transfer.
      For free convection
             q_c = hA\Delta T
             q_=heat transferred by convection
             h = convection heat transfer coefficient
             For horizontal cylinders in air h = .27 (\Delta T/L)^{.25} (Ref. 2.3, Page 2.12)
             A = Surface Area
             L = Characteristic length in feet (diameter or width)
             q_{c}=.27 (\Delta T/L)^{.25} A\Delta T
      For radiation
             q_r = sAe(T_s^4 - T_a^4)
                                                                   (Ref. 2.3, Page 2.11)
             q<sub>r</sub>= Heat transferred by radiation
             s = 1.714X10<sup>-9</sup> BTU/Hr-Ft2-R<sup>4</sup>, Boltzmann Constant
             A= Surface area
             e = Surface Emittance = .9
                                                                   (Assumption 4.1)
             T = Absolute Temperature, Rankine
            q_r = 1.714 \times 10^{-9} (.9) A (T_1^4 - T_2^4)
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For total heat transferred from the surface

$$q_3 = qc + qr$$

 $q_{a} = .27 (\Delta T/L)^{.25} A \Delta T + 1.714 X 10^{.9} (.9) A (T, -T, -T)^{.9}$

 $q_{s} = [.27 (\Delta T/L)^{.25} + 1.714 \times 10^{-9} (.9) (T_{1}^{4} - T_{2}^{4}) / \Delta T] A \Delta T$

 $\Delta T/q_s = R_s = 1/\{[.27(\Delta T/L)^{.25} + 1.714X10^{-9}(.9)(T_1^4 - T_2^4)/\Delta T]A\}$

5.4 Calculation of ACF

The ACF is calculated using a spreadsheet in accordance with the methodology described above. A description of the spreadsheet follows:

OD/W This is an input value of the conduit outside diameter or cable tray width in inches. Conduit diameters are obtained from Reference 2.5.

TH This value is the thermo-Lag thickness in inches.

ODT This is the outside diameter of the raceway with any wrap calculated from the OD and TH. For cable tray OD is not calculated because it will always be equal to W.

Α The outer surface heat transfer area. Note that for cable tray, both the top and bottom areas are included. Area is calculated on the basis of a one foot length of raceway.

Inside thermal resistance as defined above. The value is calculated from the Ri test data with no wrap in accordance with the following formula. The Ri value. calculated is then used for the cases with fire barrier installed. Note that there is no Rg and Rt for this case.

> $Ri = \Delta T/q - Rs$, Where $\Delta T = 90^{\circ}F$ (Temp drop from conductor surface to ambient)

Rg Gap thermal resistance as defined above. The value is calculated from the test data for raceway with fire barrier in accordance with the following formula. The R_a value calculated is then used for extrapolating cases with a different thickness of fire barrier.

$$Rg = \Delta T/q - (R_1 + R_2 + R_3)$$
, Where $\Delta T = 90^{\circ}F$

Thermo-Lag thermal resistance. The value is calculated in accordance with the Re following equations which were developed above.

•	Conduit	$R_{t} = Ln (ODT/OD) / 2\pi k$,	k=.1	(Ref. 2.7)
	Tray	R _t = TH/kA,	k=.1	



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 $R_s$  Surface thermal resistance is calculated in accordance with the following equations which were developed above. Note that the  $\Delta T$  in this equation is between the surface and ambient and the T<sup>4</sup> values must be in °R. The ambient temperature used is 104°F/564°R.

 $Rs = 1/[.27((T_s-104)/ODT)^{.25} + 1.714X10^{-9}(.9)((T_s+460)^{4}-564^{4})/(T_s-104)]A$ 

- $T_s$  Surface temperature of Thermo-Lag or bare conduit. The value is determined by iteration until q = q'.
- q Heat transferred For test cases, the test data is used. For extrapolated cases, it is calculated as follows:

 $q = \Delta T / (R_i + R_g + R_t + R_s)$ , Where  $\Delta T = 90^{\circ}F$ 

q' Heat transferred from the surface - Calculate heat transferred from the surface as follows:

$$q = \Delta T/R_s$$
, Where  $\Delta T = T_s - 104^{\circ}F$ 

From continuity, the heat transferred from the surface is the same as the total heat transferred. In order to solve the various cases, Ts is adjusted by iteration until q = q'.

ACF Ampacity correction factor calculated by the following equation which was developed above.

 $ACF = (q_p/q)^{1/2}$ 

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# **RACEWAY HEAT TRANSFER AND AMPACITY DE-RATING**

### - CONDUIT

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| ).939 <sup>,</sup> |
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## **RACEWAY HEAT TRANSFER AND AMPACITY DE-RATING**

| CABLE TRAY/BA                                             | NKED   |                  | *         |            |           |                                                                                   |          |        |        |        |             |
|-----------------------------------------------------------|--------|------------------|-----------|------------|-----------|-----------------------------------------------------------------------------------|----------|--------|--------|--------|-------------|
|                                                           | W      | TH               | Â         | Ri         | Rg        | Rt                                                                                | Rs       | Ts     | q      | - q'   | ACF         |
|                                                           | IN     | IN               | SQFT      | BTU/HR-F   | BTU/HR-F  | BTU/HR-F                                                                          | BTU/HR-F | F      | BTU/H  | BTU/H  |             |
| Values Extrapolate                                        | d fron | n 4 X 24" Tray ' | Test wl.: | 1 Hour Wra | ар        |                                                                                   |          |        |        |        |             |
| Test Unwrapped                                            | 24     | 0                | 4         | 0.071      | 2 . y . 2 | ઽૡૡ૽૾ૺ૾૾ૻૺ૿૿ૼૢૼ૾ૡૢૢૢૢૢ૽૾ૼૻૣ૽ૻ૱૱<br>ૡૢ૾૱ૢૢૢૢૢૢૢૢૢૢૢૢૢૢૢૢૢૢૢૢૢૢૢૢૢૡ૽ૻૢ૾૾૾ૢૢૢૢૢૢૢૢૢૢ | 0.1304   | 162.24 | 446.78 | 446.78 |             |
| Test Wrapped                                              | 24     | 0.625            | 4         | 0.071      | 0.228     | 0.1302                                                                            | 0.1492   | 127.22 | 155.67 | 155.67 | 0.590       |
| Extrapolated 1 HR                                         | 24     | 1.5 🕺 🚦          | 4         | 0.071      | 0.228     | 0.3125                                                                            | .0.1534  | 122.06 | 117.70 | 117.70 | 0.513       |
| Extrapolated 3 HR                                         | 24     | 3.06             | 4         | 0.071      | 0.228     | 0.6375                                                                            | 0.1585   | 117.03 | 82.21  | 82.21  | `∙0.429`    |
| Predict 3 HR Test                                         | 24     | 1.125            | 4         | 0.071      | 0.228     | 0.2344                                                                            | 0.1518   | 123.95 | 131.40 | 131.40 | 0.542       |
| Values Extrapolated from 4 X 24" Tray Test w/ 3 Hour Wrap |        |                  |           |            |           |                                                                                   |          | •      |        |        |             |
| Test Unwrapped                                            | 24     | 0                | 4         | 0.082      | 54        |                                                                                   | 0.1315   | 159.53 | 422.30 | 422.30 | Call States |
| Test Wrapped                                              | 24     | 1.125            | 4         | 0.082      | 0.155     | 0.2344                                                                            | 0.1503   | 125.78 | 144.88 | 144.88 | 0.586       |
| Extrapolated 1 HR                                         | 24     | 1.5              | 4         | 0.082      | 0.155     | 0.3125                                                                            | 0.1521   | 123.53 | 128.36 | 128.36 | 0.551       |
| Extrapolated 3 HR                                         | 24 -   | 3.06             | 4         | 0.082      | 0.155     | 0.6375                                                                            | 0.1577   | 117.76 | 87.23  | 87.23  | 0.454       |

| CALCULATION N | 0PSL-BFSM-98-005 | REV SHEET NO11 |  |
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|               |                  |                |  |
| Results       | *                | · ·            |  |

The most conservative results for 1 hour and 3 hour conduit and cable tray are listed below. The less conservative values from the spreadsheet can also be used for applicable field conditions.

| Item |                         | ACF |
|------|-------------------------|-----|
| 1 H  | R Conduit               | .94 |
| з н  | R Conduit               | .84 |
| 1 H  | R Tray (Banked Conduit) | .51 |
| 3 H  | R Tray (Banked Conduit) | .43 |

Note that these correction factors are contingent upon the maximum thickness, installation requirements, and size limits detailed in the Assumptions/Basis.

### Discussion

The calculation spreadsheet provided negative values for  $R_g$  for conduit. A negative value for thermal resistance has no real physical meaning. The negative value is a result of back calculating the resistance from test data. As the total resistance is made up of 4 components, the negative value is simply a correction for a resistance value that is excessive for one of the other components. The negative value does not interfere with the calculation because it is always added to the other components to obtain the total resistance.

When the methodology was used to predict the ACF for the tested 3 hour barriers using the test data from the 1 hour barriers, the results were as follows:

|                             | Predicted Value | Т | est Value |
|-----------------------------|-----------------|---|-----------|
| 4" Conduit w/3 hour barrier | .96             |   | .97       |
| Cable Tray w/3 hour barrier | . 54            | • | .59       |

These results demonstrate that the methodology used to extrapolate the test data provides conservative and reasonably accurate values.

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## THERMO-LAG® 770 FIRE BARRIER SYSTEM

### PHYSICAL AND MECHANICAL PROPERTIES

| Property             | Nominal Value                | Procedure  |
|----------------------|------------------------------|------------|
| Sprayed Density      | 62 Lbs/Fr <sup>3</sup>       | ASTM D 792 |
| Hardness             | 50                           | Shore D    |
| Thermal Conductivity | <u>il Btu-Ét</u><br>Fultir F | ASTM C 177 |
| Tensile Strength     | 850 psi                      | ASTM D 638 |
| Compressive Strength | . 625 psi                    | ASTM D 695 |
| Flexural Strength    | 2500 psi                     | ASTM D 790 |
| Flexural Stiffness   | 90 ksi                       | ASTM D 790 |
| Bond Strength        | 700 psi @ RT                 | ASTM D 952 |
| Initial Modulus      | 75,000 psi                   | ASTM D 638 |
| Shear Strength       | 1,250 psi                    | ASTM D 732 |
|                      | •                            |            |

For additional information, consult the THERMO-LAG 770 Subliming Material data sheet.

THERMAL SCIENCE, INC. 2200 Cassens Drive St. Louis, Missouri 63026 Tele: (314) 349-1233 Fax: (314) 349-1207

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