



Arizona Nuclear Power Project

DOCUMENT NUMBER

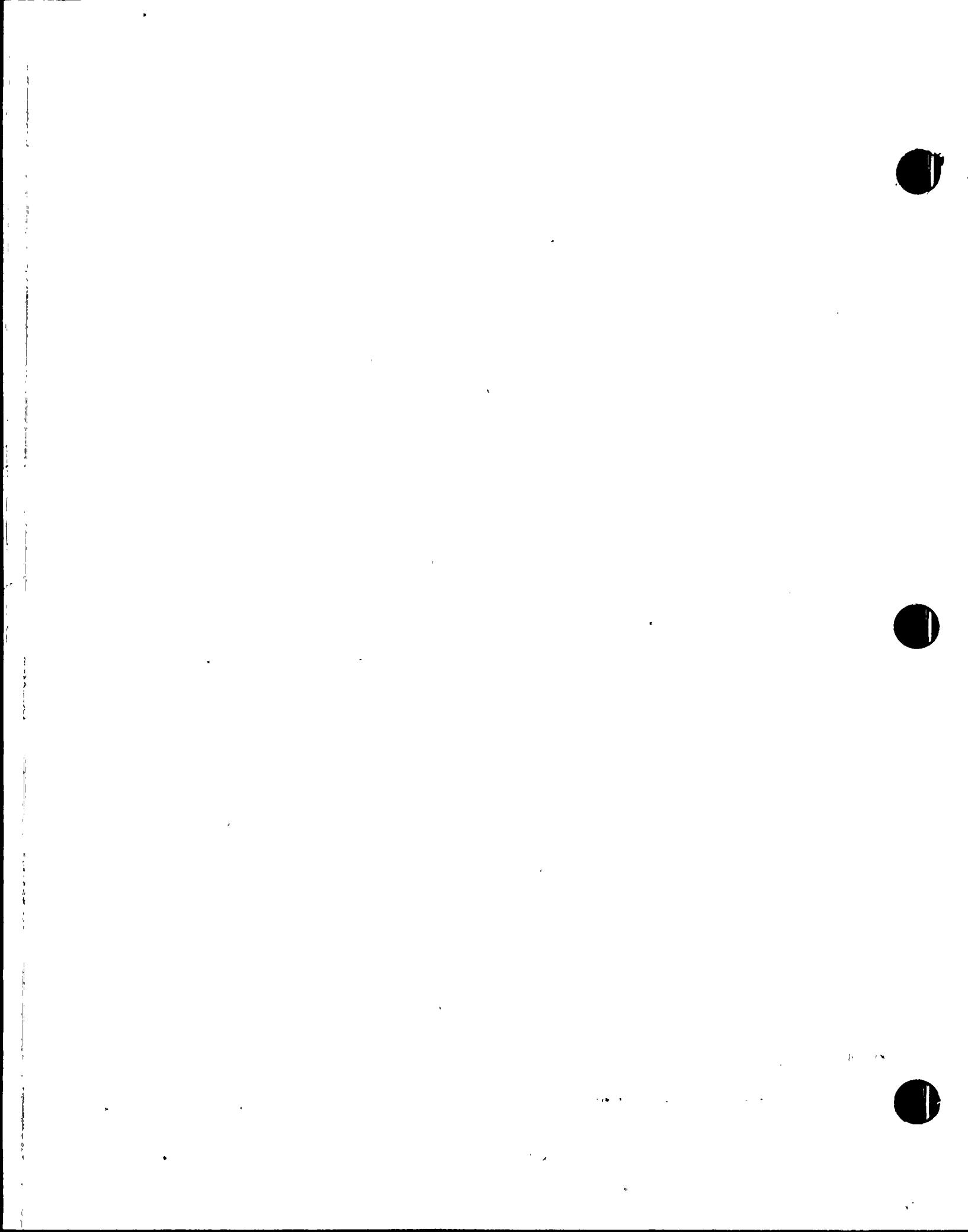
13-MC-HJ-253

TITLE/DESCRIPTION

TRANSIENT TEMPERATURE STUDY  
FOR MAIN CONTROL ROOM

QUALITY CLASS Q  
SAFETY RELATED

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△	ISSUED FOR USE	BECHTEL	BECHTEL	Denny Riley	N/A	N/A	
REV	REVISION DESCRIPTION	ORIGINATOR DATE	CHECKER DATE	RS DATE	QA DATE	NEM/PEM DATE	





INTERNAL CONTROL NO.

18601 - 200 - CALC - 001

## CALCULATION COVER SHEET

SHEET 1

PROJECT

ANPP PVNGS

JOB NO. 18601-200

CALC. NO. 13-MC-H.I-25

SUBJECT

TRANSIENT TEMPERATURE STUDY FOR THE MAIN CONTROL FILE NO. \_\_\_\_\_

ROOM

PROJECT  
QUALITY CLASS

QR

DISCIPLINE MECH

COMPUTER  
PROGRAM SCP  
 YES     NOPROGRAM NO.(S)  
ME-204VERSION/RELEASE NO.  
A1

## RECORD OF ISSUES

NO.	DESCRIPTION	TOTAL NO. OF SHEETS	LAST SHEET NO.	ORIG	CKR	CL	GS	CHEF	DATE
0	ORIGINAL ISSUE	51	51	PSS	JJ	-	JCB	-	APPR. 6/13/84 FILM
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## INFORMATION ENTERED IN THIS SPACE:

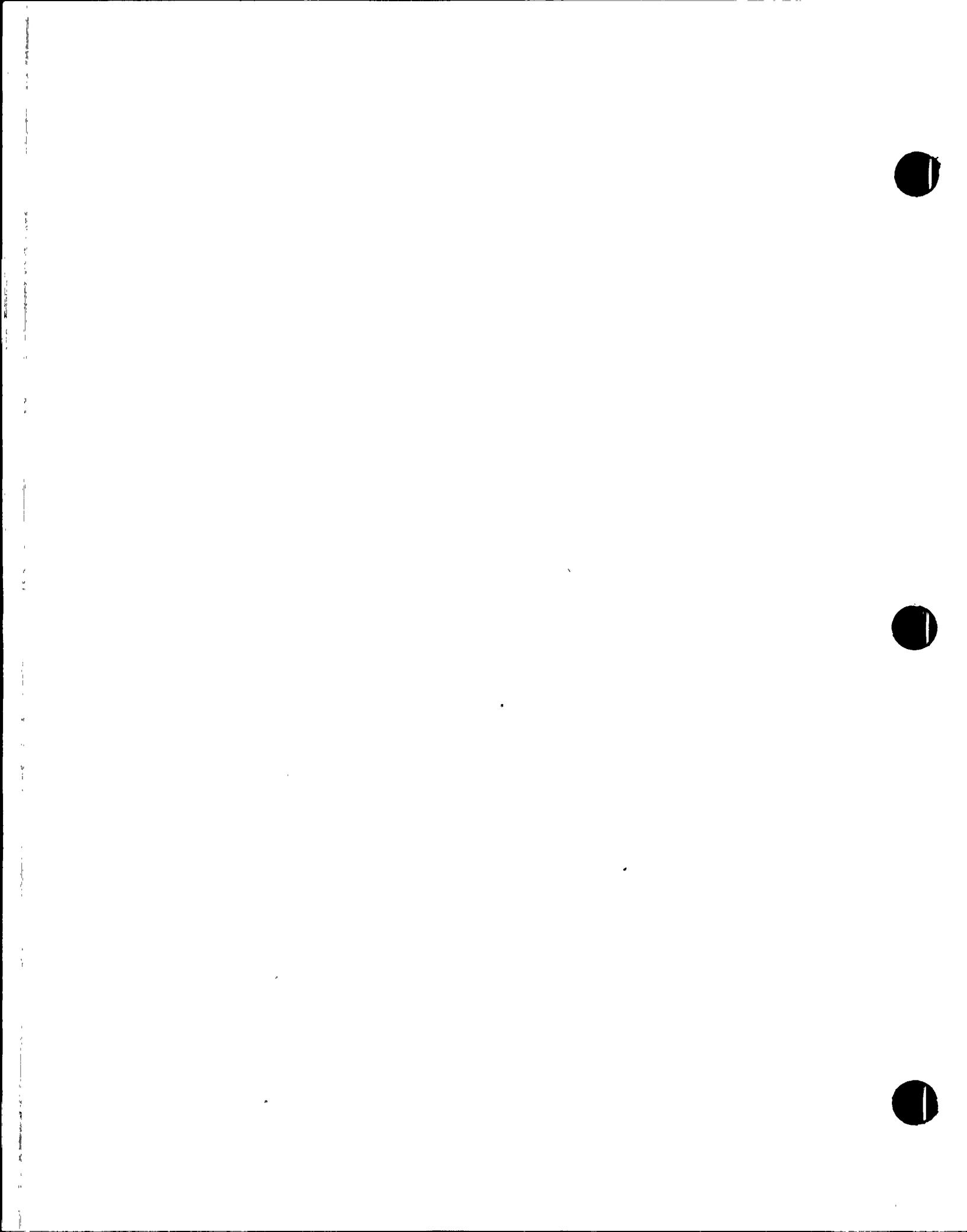
- SHOW PROFESSIONAL ENGINEER STAMP, IF REQUIRED.
- ENTER REFERENCE TO INCLUSION OF CHECKER'S ALTERNATE CALCULATIONS, IF USED.
- PROVIDE ANY NOTES TO ASSIST CHECKING AND APPROVAL

Utilization of these calculations by persons without access to pertinent factors and without proper regard for their purpose could lead to erroneous conclusions. Bechtel cannot assume responsibility for the use of these calculations not under its direct control.

## NOTICE

APS ACKNOWLEDGES THAT THESE DESIGN CALCULATIONS ARE ONLY AN ISOLATED PART OF THE COMPLETE DESIGN FOR THE SYSTEM THEY CONCERN, AND ARE SUBJECT TO BEING TAKEN OUT OF CONTEXT, MISINTERPRETED OR MISCONSTRUED IF USED WITHOUT BECHTEL POWER CORPORATION'S DIRECT PARTICIPATION.

PF-6346 (10407) 2/84





# CALCULATION SHEET

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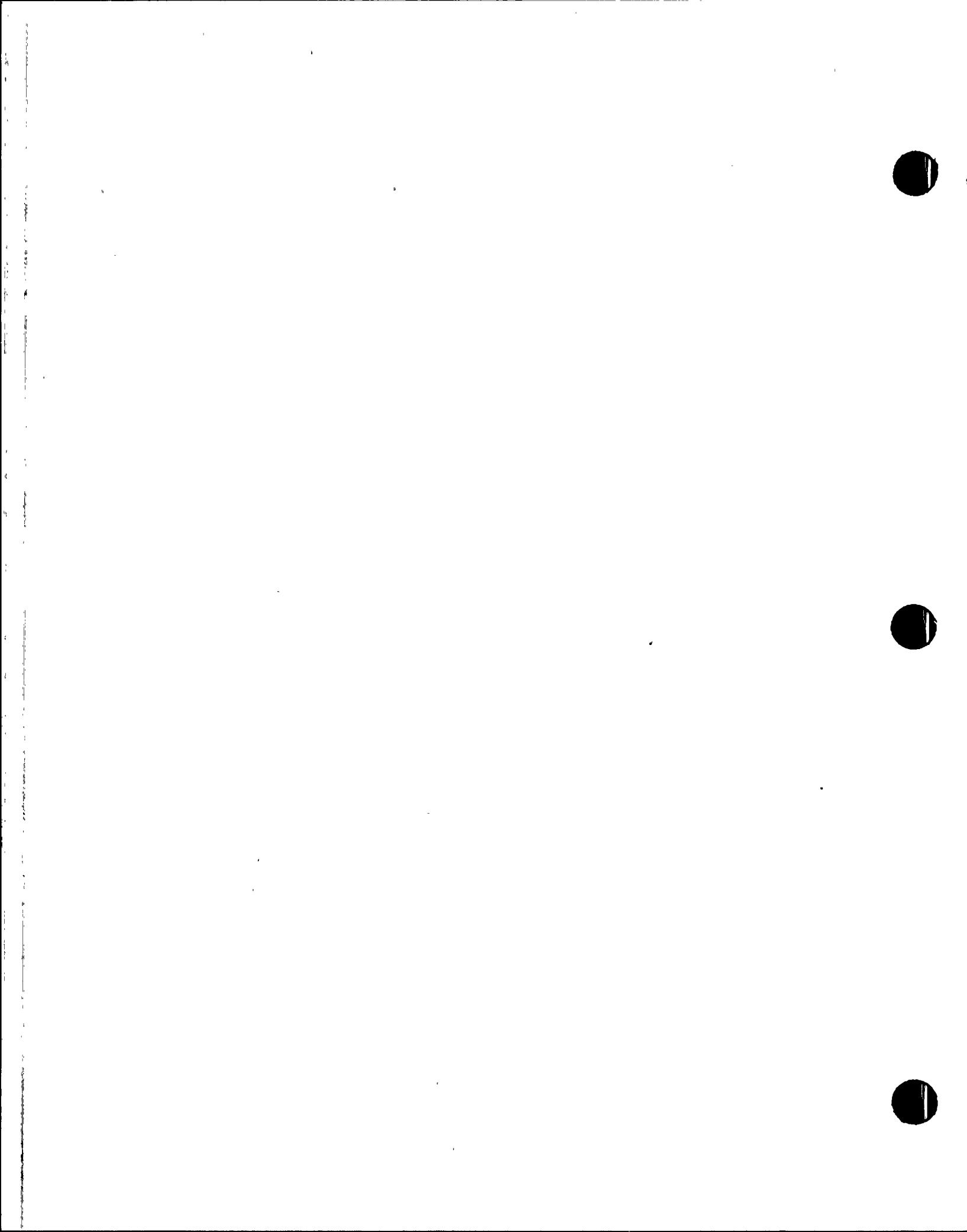
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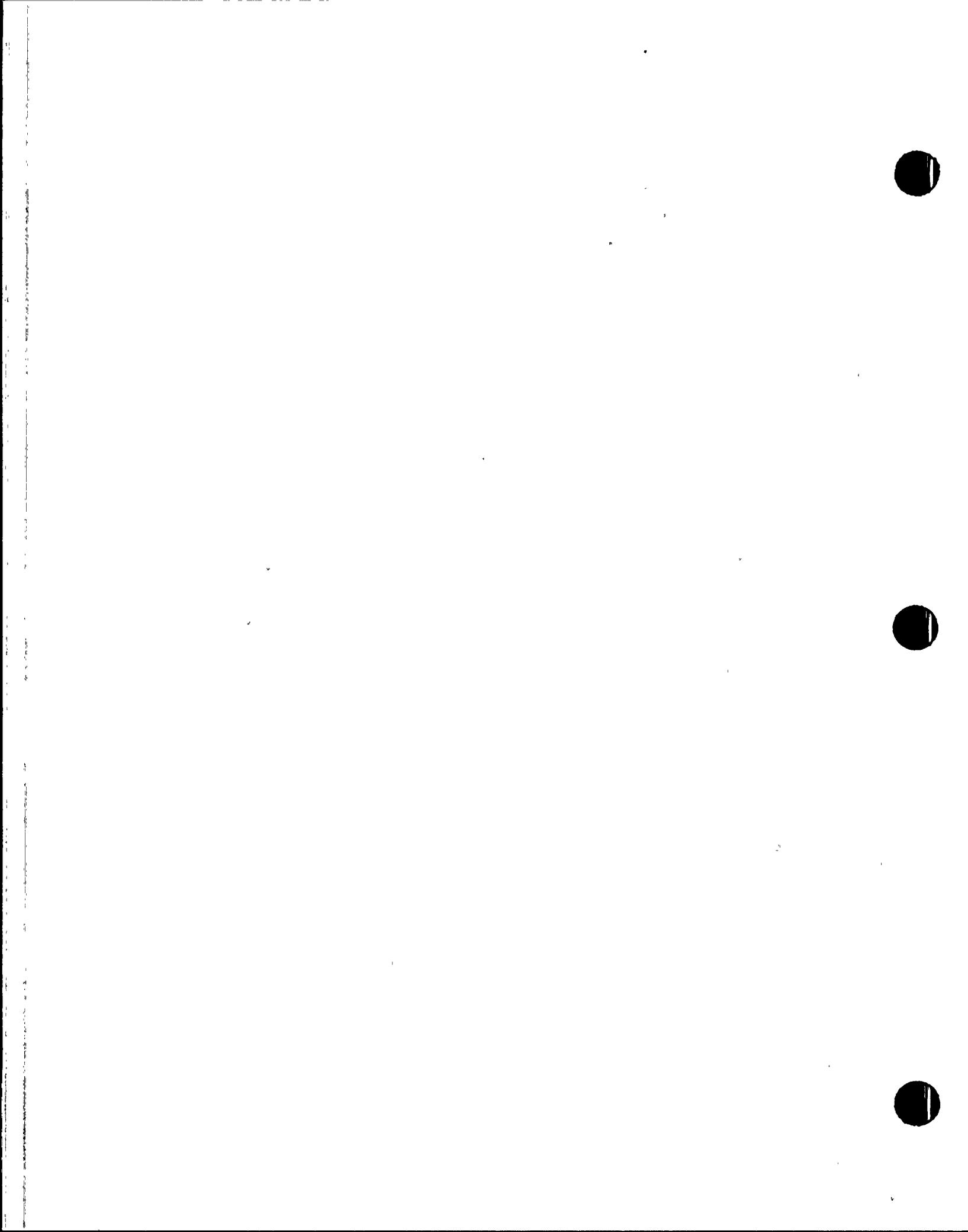
I. PURPOSE

The design basis for cooling the Main Control Room Complex uses Essential Air Handling Unit (AHU) and the Essential Chilled Water System (ECWS) for cooling. In the event that either the AHU or ECWS fails, the room temperature will rise rapidly. The purpose of this study is to determine the transient air temperature for 24 hours in the Main Control Room Complex for the following case:

- o Large break LOCA
- o Normal offsite power is available and all normally operating equipment continues to generate heat
- o Normal HVAC stops and does not operate
- o Essential chilled water system is not operating
- o Essential air handlers/fans are operating
- o Doors and other HVAC barriers remain in their normal closed position during the entire period of the study

Standard room heat-up (RMHTUP) computer program, ME 204, version A1, is used to study the room air heat-up by the equipment and other heat loads in the Main Control Room Complex.

The heat generated in the room is transferred to the room ambient air, stored in the room enclosure concrete (heat sink) and transferred to the air outside the room. The transient temperature for the Main Control Room Complex is calculated for a time period of 24 hours for the above mentioned conditions. The emphasis of this study is to estimate the room air temperature more accurately in the first 1 1/2 hours rather than the later part of the problem.



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## CALCULATION SHEET

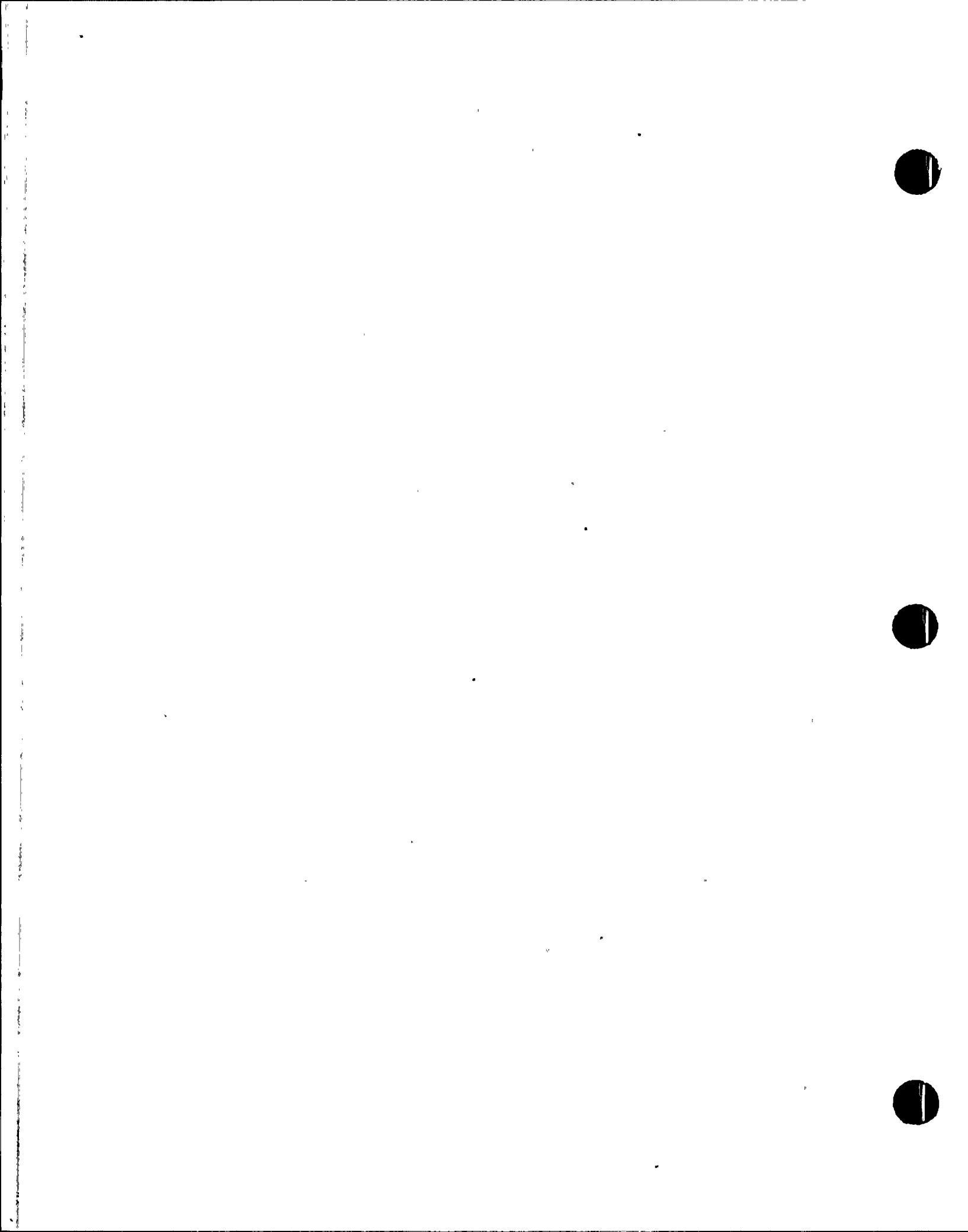
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II. DESIGN CRITERIA:

This is a study of the effect of certain equipment failures which are, strictly speaking, beyond the specific design basis for the affected systems. The results will be used as input for a response to EER 88-EC-018 being done by others.

The systems involved in this study are the control room HVAC (Normal and Essential), and Essential Chilled Water. Their design criteria are references 10 and 11.





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5. The main control room complex model is simplified as an enclosed space bounded by the same thickness of walls, ceiling and floor. Most of the room walls are 1'-9" thick (Ref 5). The floor and the walls around the cable shafts are 1'-0" thick (Ref 5), while the ceiling is 11" thick, (Ref 6). For this study, all surfaces are considered to be 1'-0" thick. No credit is taken for the wall thickness being more than 1'-0", except that the ceiling is also considered as 1'-0" thick for computer modeling.

The emphasis in this calculation is to improve the accuracy of the calculation in the initial 90 minutes. Thus, heat sinks are modeled in a way that takes credit for thinner heat sink walls than would be used in modeling for accuracy in the 12 to 24 hour time frame.

Due to the low heat load generated during the first 1 1/2 hours compared to the thermal capacity of the existing walls to absorb the heat, additional thickness does not appreciably affect the room ambient air temperature.

6. The walls, ceiling and floor are used as a heat sink. For details, refer to Section VI.C of this study.

7. The heat generated within the room is considered as being constant. This is due to the fact that the heat load equipment continues to operate during this study. In addition, a constant heat load is required for the computer model as shown in section VI.D.





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8. The dimensions of the Main Control Room Complex for all three units are essentially identical. In addition, the chillers, fans, other equipment, the lighting and the room dimensions are virtually the same for all of these rooms. This study is performed for the Main Control Room Complex for Unit 1 and is applicable to the Main Control Rooms for all three units.
9. Per assumption 3 above, the essential chilled water is not available during the time period of this study. For a conservative approach, the AHU fans are assumed running, without chilled water through the essential coils as indicated in Section I.
10. This study is performed for a time period of 24 hours. The computer model can provide details for a maximum of 720 steps (Refer Section VI.D). Therefore, each step or time increment is 2 minutes.
11. The room walls are required to be divided into a number of layers for computation of temperature distribution in the concrete walls by the computer model. The required input for the thickness of the first layer and the multiplication factor for thickness of other layers are selected as 0.01 ft (approximately 1/8") and 1.41 respectively. For details, see sections VI.A and B.





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2  
3 12. Concrete has the following properties:

- 4 A. Density 144 lbs/ft<sup>3</sup> (Ref 12, table A-2) *SL/HK/6/88*
- 5 B. Thermal conductivity 0.54 (Ref 12, table A-2)
- 6 C. Specific heat 0.2 btu/lb-°F (Ref 12, table A-2)

7  
8 13. The net Control Room Complex volume is required as input to  
9 the computer model. For calculation of net room volume,  
10 the volume of the HVAC equipment, the control panels and  
11 other electrical equipment in the Main Control Room Complex  
12 is insignificant and is neglected.

13  
14 Most equipment in the main control room complex is control  
15 panels and instrument racks. These items are mostly air  
16 (i.e. approximately 90+% of the gross volume occupied by  
17 the panel is air. Much less than 10% is actually metal,  
18 wire, circuit boards, terminal boards, etc.).

19  
20 In addition, the air has low thermal capacity. A small  
21 difference in the room volume does not have significant  
22 effect on the room temperature as shown by volume  
23 sensitivity analysis Case B of Reference 17.

24  
25 14. All miscellaneous metal items (e.g. structural steel,  
26 platforms, cold piping) which could be considered as heat  
27 sinks are conservatively ignored. This is due to the  
28 computer program limitation which allows for a single heat  
29 sink to be included in the model.

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## CALCULATION SHEET

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3 15. At the start of this study, both trains of the Essential  
4 Control Room Complex HVAC will be started. For calculation  
5 of heat loads, both of these trains continue to operate  
6 during the entire period of this calculation. This  
7 assumption complies with Section I.  
8
- 9 16. The temperature of the outside makeup air is  $101.3^{\circ}\text{F}$ . This  
10 is based on the actual measurements made by ANPP at the  
11 operating unit (Ref. 16). The walls around the outside air  
12 chase are neglected for calculation of the heat sink  
13 effect. These walls effectively act as a heat source  
14 during initial period but act as a heat sink during the  
15 later period. The amount of heat added during the initial  
16 stages is insignificant because the structural steel and  
17 the thin metal items (e.g. cabinet sheet metal) act as a  
18 heat sink, the effect of which has been excluded (see  
19 Assumption 14 and Section VII.F).  
20
- 21 17. Room surface area adjustment for panels etc. mounted on the  
22 concrete floor and walls is not done because the panels,  
23 instrument racks, etc. generate heat internally and  
24 generally have open bottoms which allow the heat to have  
25 ready access to the heat sink.  
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## CALCULATION SHEET

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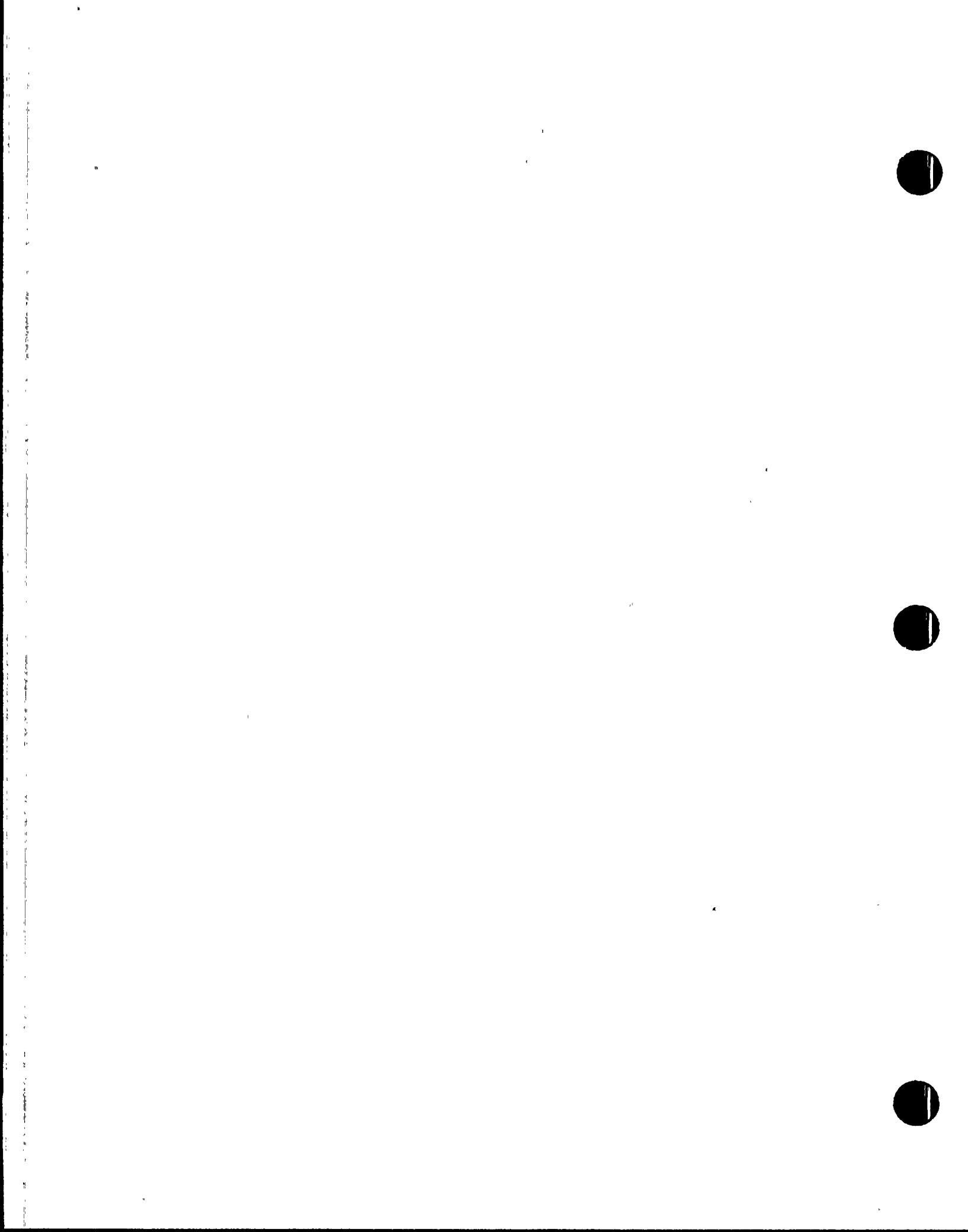
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REV  
INDEX  
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## IV. REFERENCES:

1. ASHRAE Handbook of Fundamentals, 1972..
2. Control Room Complex - Heat Load Calculations, Calculation number 13-MC-HJ-051, Rev. 2.
3. P & I diagram, Control Building HVAC, drawing number 01-M-HJP-001, Rev. 12.
4. Basic Flow Diagram, HVAC Control Room - Control Building, drawing number 13-M-HJF-001, Rev. 3
5. Control Building Plan at Elevation 140'-0", drawing number 13-C-ZJS-140, Rev. 19.
6. Control Building Plan at Elevation 160'-0", drawing number 13-C-ZJS-150, Rev..14.
7. Control Building, Control Room Plan and Interior Elevations, drawing number 13-A-ZJD-503, Rev. 14:
8. User's and Theoretical Manuals Verification Report, program RMHTUP-Room Heat Up, program number ME.204, version A1, Bechtel Power Corporation, San Francisco Power Division.
9. Updated Final Safety Analysis Report, Palo Verde Nuclear Generating Station, Revision 0.
10. Detailed Design Criteria, Part III, System HJ, HVAC Control Building, Rev.5:
11. Detailed Design Criteria, Part III, System EC, Essential Chilled Water System, Rev. .3.
12. Principles of heat transfer, third edition, Frank Kreith, Intext Education Publishers, table A-1, Properties of Metals and Alloys, page 634 and table A-2, Physical Properties of Some Nonmetals, page 635.
13. Operation and Maintenance Manual for Control Room Essential Air Handling Units, CTI Nuclear Inc., SDR Log M721B-582-4



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## CALCULATION SHEET

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0	PSS	6-13-88	Morris	9/13/88	1				
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14. Control Room Complex Essential Cooling System Equipment Sizing Calculations, Number 13-MC-HJ-251, Rev 2.
15. Project General Design Criteria, Part II, Rev. 19.
16. Actual Temperature Measurement Data by ANPP for an operating unit, June 10, 1988 (3 pages attached).
17. Transient Temperature Study for Essential Cooling Water Pump Rooms, Calculation Number 13-MC-HA-253, Rev. 0.





## CALCULATION SHEET

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SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 13

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REV.  
INDIC.V. SUMMARY OF RESULTS:

The calculated transient room temperature summary for the Main Control Room Complex is shown below. The transient room air temperature for the Main Control Room Complex obtained from the computer output, Appendix A, shows that with both essential AHU fans of Trains A & B operating under conditions outlined in Section I, the calculated temperature in the pump room will be as shown below. For cases with one or no fans running, the transient temperature outputs, Appendices B and C, are also presented:

Time Period	Appendix A Control Room w/2 ESF Fans Temp. °F	Appendix B Control Room w/1 ESF Fan Temp. °F	Appendix C Control Room w/No ESF Fans Temp. °F
0 min	75.5	75.5	75.5
2 min	86.07	83.79	81.49
4 min	93.65	89.78	85.86
6 min	99.00	94.06	89.02
12 min	107.43	100.95	94.23
36 min	115.21	107.38	99.19
60 min	118.50	110.00	101.12
84 min	121.05	112.03	102.60
96 min	122.18	112.92	103.25
2 hr	124.21	114.52	104.42
4 hr	132.05	120.73	108.95
6 hr	138.02	125.45	112.38
8 hr	143.03	129.40	115.26
12 hr	151.46	136.06	120.10
24 hr	171.00	151.48	131.31





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SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

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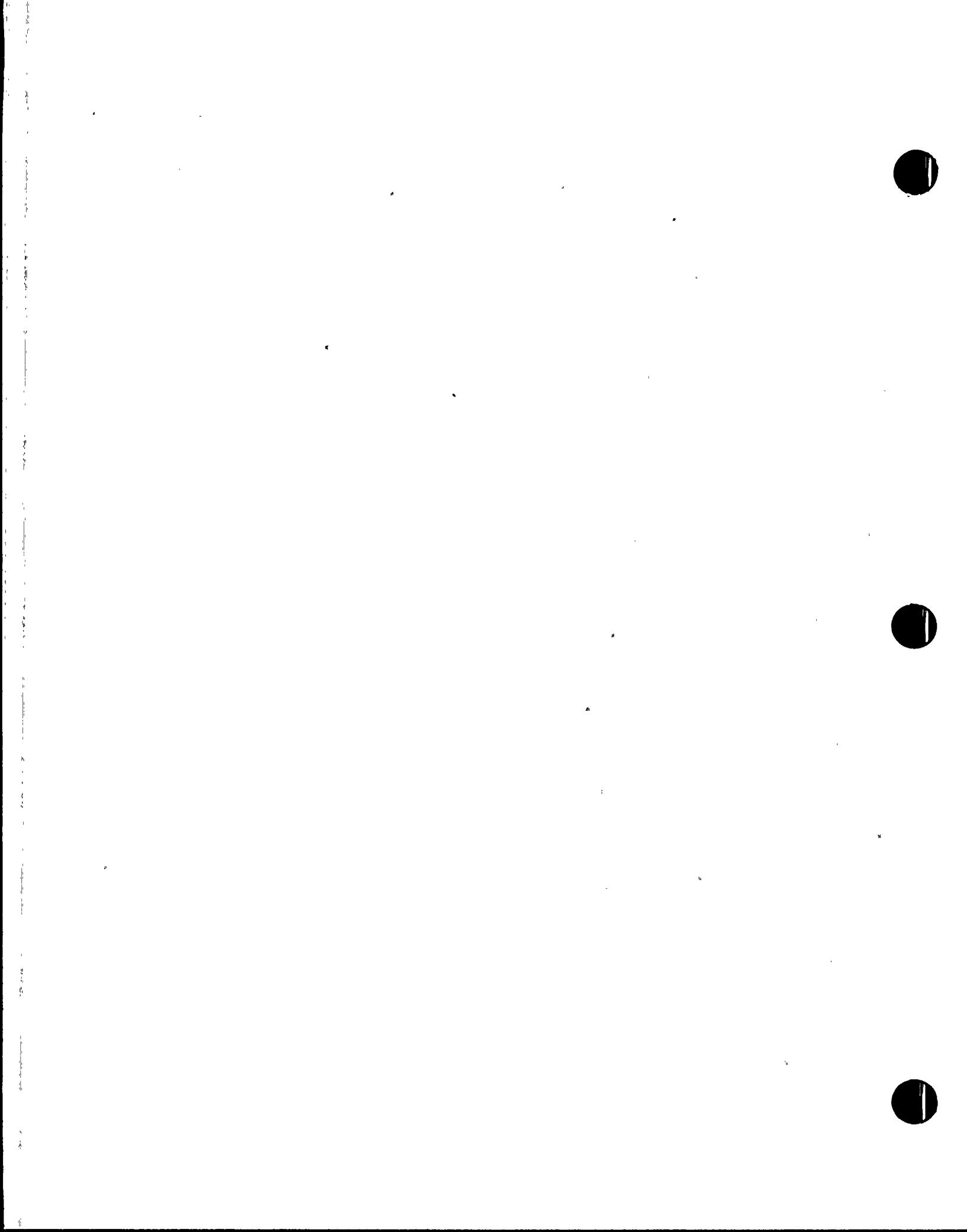
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It is readily apparent that the ESF AHU fan motors (100 bhp each) dramatically affect the control room temperature.

Stopping at least one fan quickly should be a high priority.

As shown in section VII.F, after the control room temperature exceeds the outside makeup air temperature, the control room temperature is slightly over estimated because the ventilation cooling effect of the 1000 CFM per train flowing through the control room from the control room pressurization is neglected.





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## CALCULATION SHEET

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▼VI. ROOM HEAT-UP COMPUTER MODEL

## A. Model Description: (Reference 8)

The room heat-up computer program, RMHTUP, program number ME 204, version A1, can be used to study the room ambient air heat-up by the equipment heat or any other heat sources in the room.

The temperature of room ambient air increases with time, due to the heat released from the equipment and other sources. The heat generated within the room is transferred to the ambient room air, stored in the room enclosure (walls, ceiling, and floor) and transferred to the air outside the room.

The room walls are divided into a number of layers with incremental thicknesses for numerical computation by the computer program. In this study, the value of imaginary thickness of first layer of concrete wall is selected as 0.01 ft and the multiplication factor of imaginary thickness of other layers is taken as 1.41. In other words, the first concrete layer thickness is 0.01 ft (approximately 1/8"), the second layer thickness is 0.0141 ft (approximately 3/16"), third layer  $0.01 \times (1.41)^2$  ft, etc.

The transient room temperature is determined from the heat balance equation, which balances the heat generated within the room and the heat transferred to the ambient room air, stored in the room enclosure and transferred to the outside air, as shown below:





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PROJECT ANPP

## CALCULATION SHEET

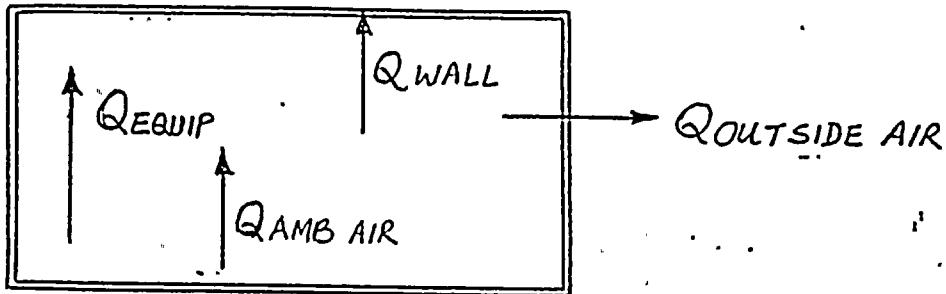
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$$Q_{EQUIP} = Q_{AMB\ AIR} + Q_{WALL} + Q_{OUTSIDE\ AIR}$$

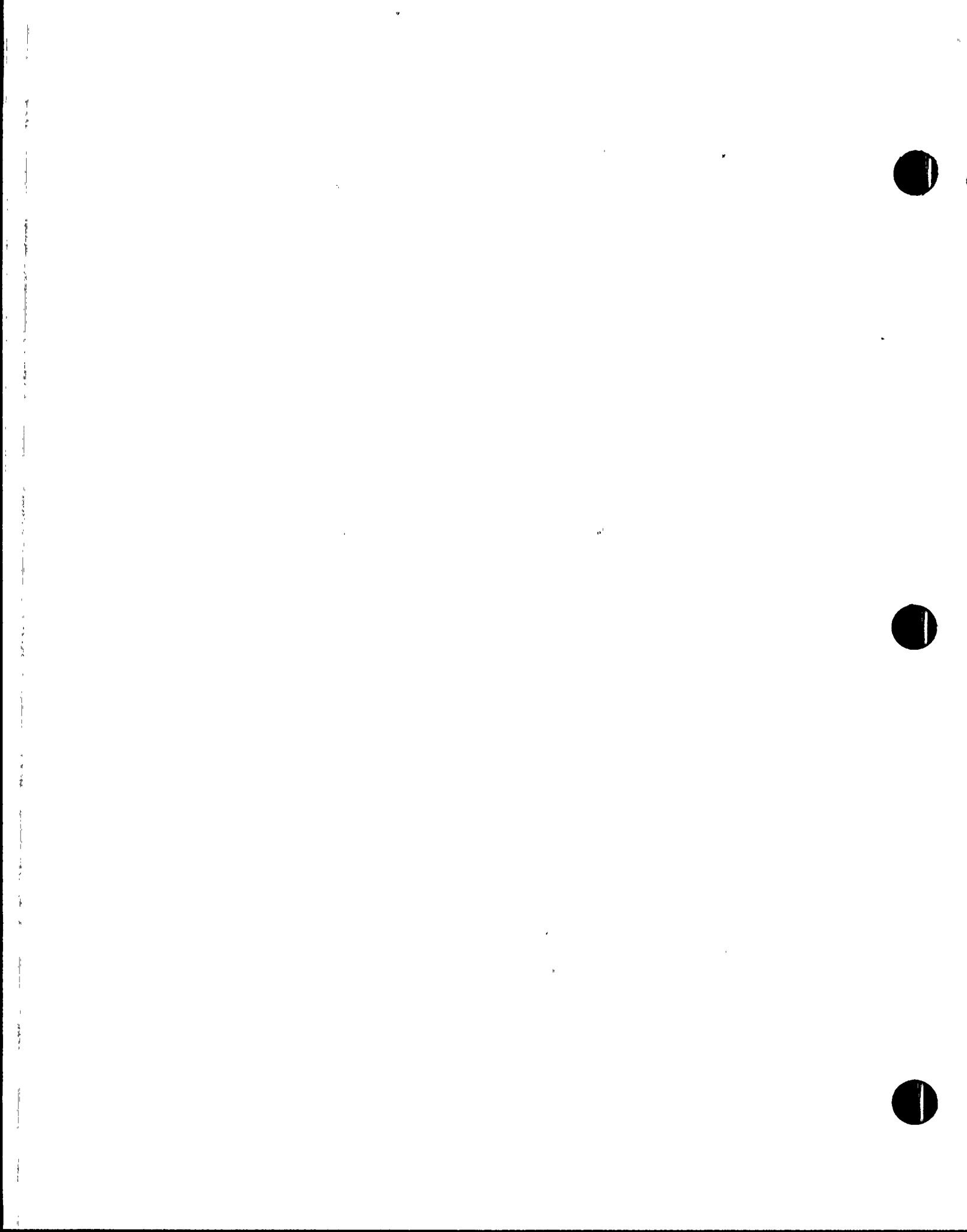
WHERE  $Q_{EQUIP}$  = Heat generated from equipment or any other source in the room, BTU/HR

$Q_{AMB\ AIR}$  = Heat transferred to the ambient room air, BTU/HR

$Q_{WALL}$  = Heat stored in room walls, ceiling and floor, BTU/HR.

$Q_{OUTSIDE\ AIR}$  = Heat transferred to the outside air, BTU/HR

The room ambient temperature and the wall temperature distribution are calculated at fixed time intervals. In addition, total heat stored in the ambient air and in the concrete walls and heat transferred to the outside air are provided for each time interval.



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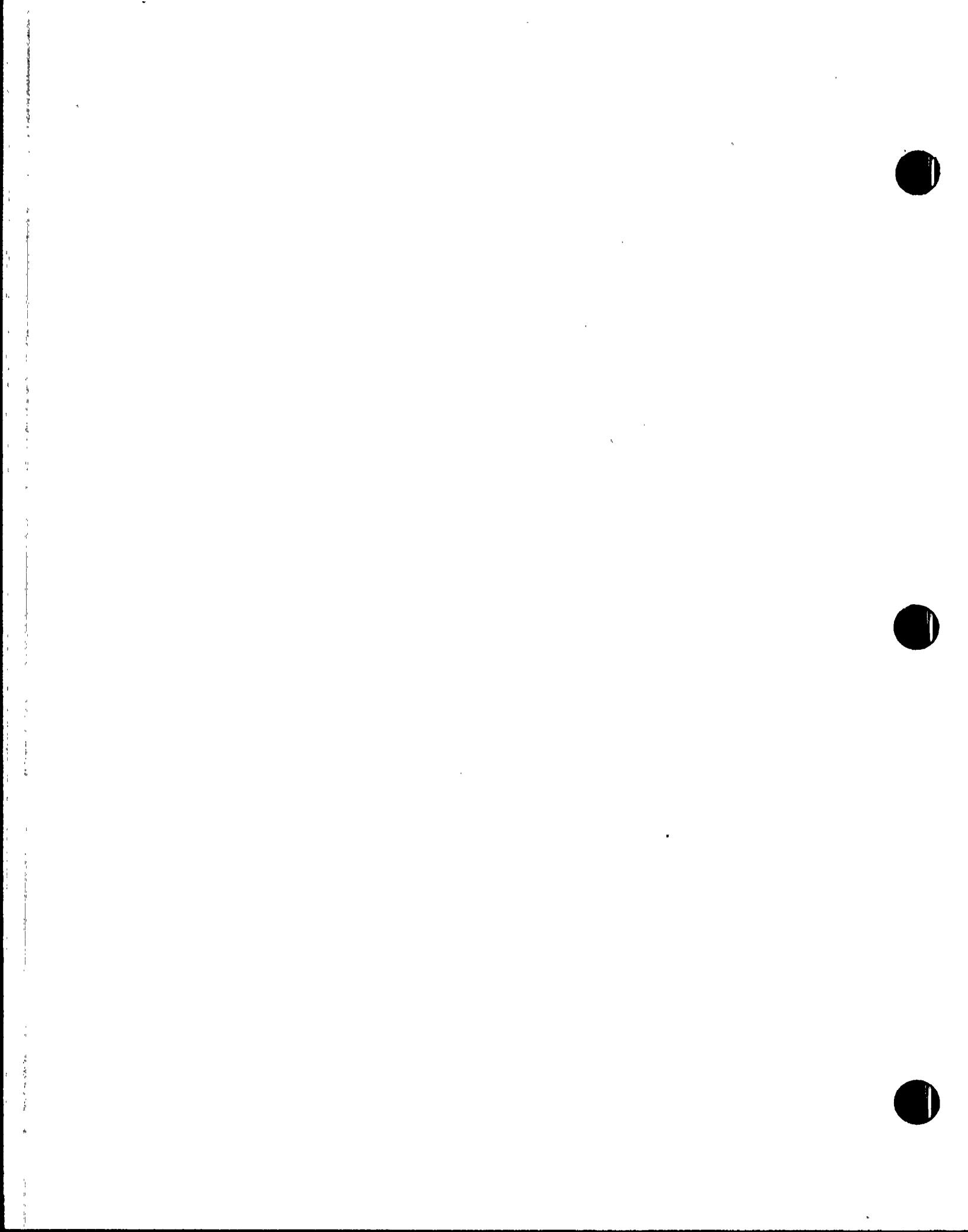
## CALCULATION SHEET

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▼3      B. Input Required for the Computer Model:  
45      The following information is required as input for transient  
6      temperature study in the Main Control Room Complex:  
7

- 8      1. Initial room ambient temperature, degrees F
- 9      2. Initial outside ambient temperature, degrees F
- 10     3. Equipment and other heat generated in the room, BTU/HR
- 11     4. Net room surface area, ft<sup>2</sup>
- 12     5. Net room volume, ft<sup>3</sup>
- 13     6. Thickness of room enclosure, ft
- 14     7. Density of room enclosure material, lbs/ft<sup>3</sup>
- 15     8. Thermal conductivity of room material, BTU/HR-ft-F
- 16     9. Specific heat of room enclosure material, BTU/lb-F
- 17     10. One period of time increment for calculation, min
- 18     11. Imaginary thickness of first layer of room enclosure, ft
- 19     12. Multiplication factor of imaginary thickness of other  
20        layers





## CALCULATION SHEET

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SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 18

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REV.  
INDIC.1 C. Assumptions for the Computer Model  
2

- 3 1. The room is simplified as an enclosed space bounded by the
- 4 same thickness of walls, ceiling, and floor.
- 5 2. The gross room volume is corrected for the volume occupied
- 6 by piping and equipment (see Assumption 13). "
- 7 3. The gross room surface area of the room enclosure is
- 8 corrected for the area occupied by equipment (see
- 9 Assumption 17).
- 10 4. The enclosure walls, ceiling and floor are taken as a heat
- 11 sink.
- 12

13 D. Computer Model Limitations  
14

- 15 1. The room enclosure walls, ceiling and floor must be
- 16 considered as having the same thickness and of the same
- 17 homogeneous material.
- 18 2. The air temperatures outside the room must be considered as
- 19 being the same and remaining constant.
- 20 3. The heat generated within the room must be considered as
- 21 being constant. No heat generated outside the room can be
- 22 considered.
- 23 4. The program is limited to 720 time period calculations.
- 24

25 E. Computer Model Output  
26

27 The following information is provided in the computer model  
28 output:

29 TAF = Final room air temperature at each period, °F

30 QAT = Heat stored in the ambient air, BTU

31 QST = Total heat stored in the concrete, BTU

32 QOT = Heat transferred to the outside air, BTU

33  
34  
35  
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SHEET NO. 19

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REV.  
INDICATOR  
▼1 VII. STUDY:

2 This study is performed for the following case:

3 Control Room Complex operates at the design flow. Normal  
4 HVAC and Essential Chillers are not available. However,  
5 both essential air handling unit fans are operating. "

## 6 A. Heat Loads:

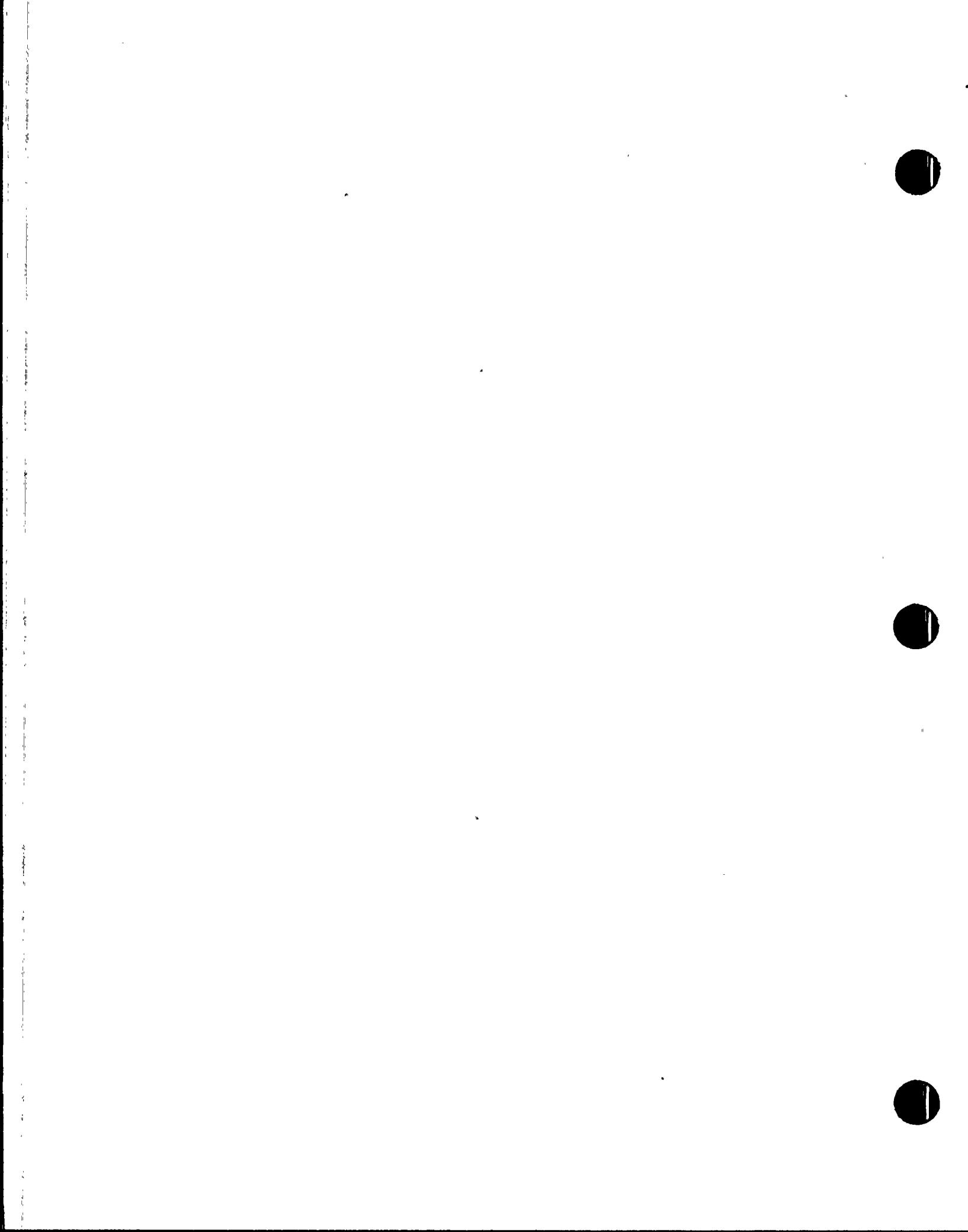
7 Design basis cooling for the Control Room Complex is  
8 provided by the Essential Air Handling Unit HJA-F04 or  
9 HJB-F04. Essential chilled water is recirculated through  
10 the AHU cooling coils. In the event that either the  
11 essential AHU or the essential chilled water system fails,  
12 the control room temperature will rise rapidly.

## 13 1. Essential AHU Fan Motors Heat Load:

14 This heat load consists of the heat generated by the  
15 Air Handling Units (HJA-F04 and HJB-F04) vaneaxial fan  
16 motors. Both essential AHU fans are running without  
17 the essential chilled water through the essential AHU  
18 cooling coils (See Assumption 9).19 Essential AHU Fan Motor BHP = 100 hp (Ref. 13)  
20 (125 hp nameplate)

21 Motor efficiency at full load = 92.9% (Ref. 13)

22 All the fan motor energy is transmitted to the Control  
23 Room Complex, although the motor is physically located,  
24 at floor elevation 74'-0". However, it is located  
25 entirely within the recirculated air stream.





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-253

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 20

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
10	PSS	6-13-88	J. Morris	6/13/88	1				

REV INDICATOR

From ref 1, chap 22, table 30, page 417,

Heat gain from Essential AHU fan motor = Bhp X 2545

% eff

$$= \frac{100 \times 2545}{0.929}$$

$$= 273,950 \text{ BTU/HR}$$

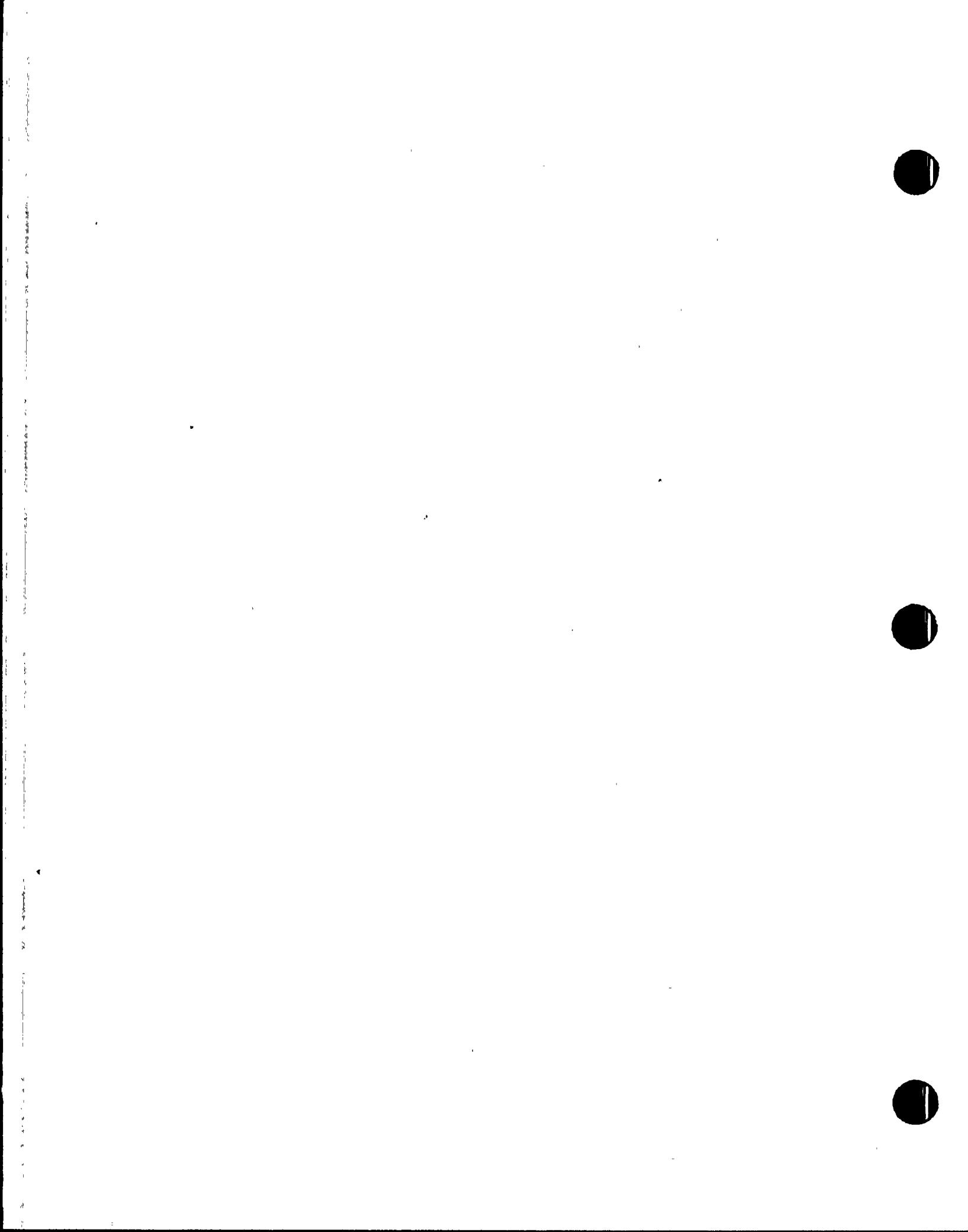
AHU Fan motors for both essential trains are in operation during time period of this study (Refer to assumption 15).

Therefore, essential AHU Fan motors heat load for Trains A & B  
=  $2 \times 273,950 \text{ BTU/HR} = 547,900 \text{ BTU/HR}$

## 2. Main Control Room Complex Heat Loads:

The Control Room normal HVAC and essential cooling system are designed to maintain the required "environment" for personnel occupancy and equipment operation during normal and emergency operations.

The heat loads in the Main Control Room Complex are different during normal and emergency conditions. The following Table 1 compares the heat loads during these two situations. The greater of each load is selected to satisfy the scenario of Section I. These heat loads include the lighting and personnel occupancy in the Main Control Room Complex and are taken from pages 6 and 7 of Reference 2.

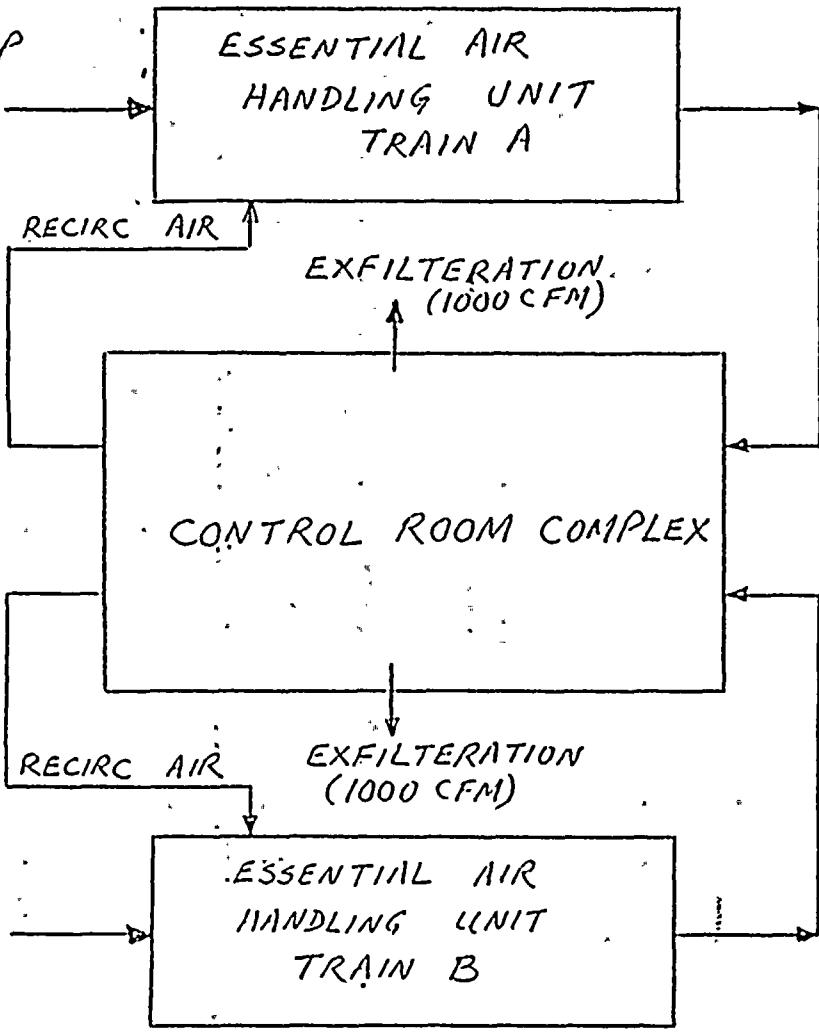


PROJECT ANPPJOB NO. 18601 - 200CALC. NO. 13-MC-HJ-25

## CALCULATION SHEET

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOMSHEET NO. 21

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
/0	PS5	6-13-86	Morin	6/13/86	/A				

OUTSIDE MAKEUP  
AIR (1000 CFM @  
 $101.3^{\circ}\text{F}$ )SYSTEM LAYOUT

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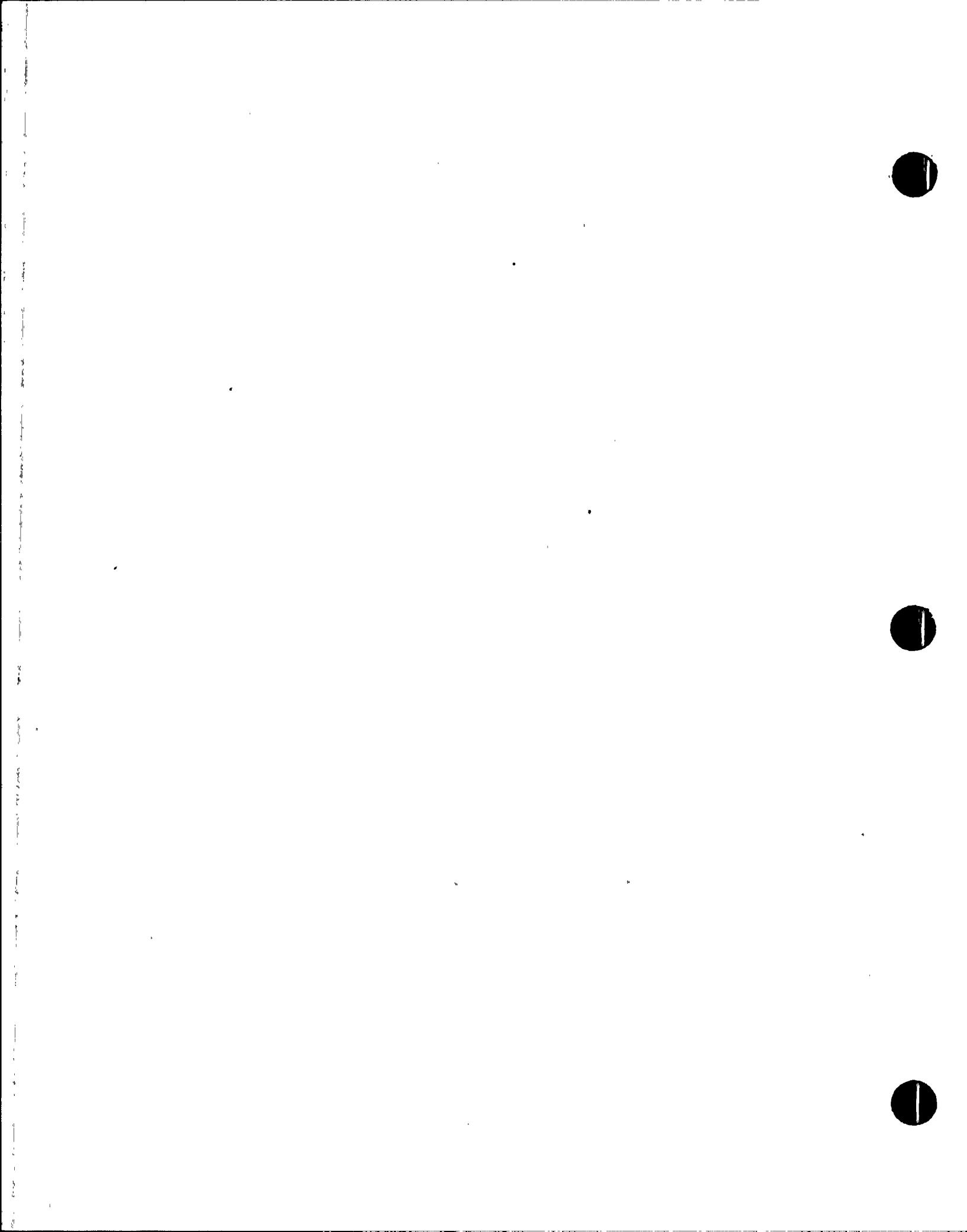
5

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PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-253

## CALCULATION SHEET

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 22

REV.	ORIGINATOR	DATE	CHECKER	DATE	REV.	ORIGINATOR	DATE	CHECKER	DATE
1	PSS	6-13-68	Thoma	6/3/68	1				
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36					36				

TABLE 1

## CONTROL ROOM COMPLEX HEAT LOADS

	NORMAL OPERATION HEAT LOAD BTU/HR	ESSENTIAL OPERATION HEAT LOAD BTU/HR	MAX OF NORMAL/ESSENTIAL OPERATION HEAT LOAD BTU/HR
OPERATOR CONSOLE	77,127	64,385	77,127
CABINET AREA	358,185	280,000	358,185
COMPUTER ROOM	90,110	90,110	90,110
CORRIDOR	7,195	7,195	7,195
SHIFT SUPERVISOR	5,775	5,775	5,775
KITCHEN & PANTRY	9,660	9,660	9,660
OFFICE #1	4,200	4,200	4,200
OFFICE #2 & 3	13,300	13,300	13,300
INSTRUMENT REPAIR	16,000	16,000	16,000
CONFERENCE ROOM	13,875	13,875	13,875
MEN'S TOILET	2,335	2,335	2,335
WOMEN'S TOILET	2,085	2,085	2,085
RETURN AIR PLENUM	68,028	68,028	68,028
COMMUNICATIONS 120' LEVEL	22,500		22,500
SUBTOTAL	690,375	576,948	690,375
MARGIN		35,052	
TOTAL	690,375	612,000	690,375

- NOTE: 1. The heat load data is taken from the calculation 13-MC-HJ-051, Rev. 2 (Reference 2).  
 2. The lighting loads are included in these heat loads.  
 3. The personnel occupancy heat loads are included in the above shown heat loads.

REV.  
INDICATOR





20mcr

PROJECT ANPP

## CALCULATION SHEET

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-25

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO.

23

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-88	Yours	6/13/88	1				
1					2				

## 3. Total Heat Loads:

4. Summary of the heat loads in the Control Room Complex:

6. Essential AHU fan motors heat load

7. for Train A &amp; B = 547,900 BTU/HR [see section VII.A.1]

8. Main Control Room Complex

9. heat load = 690,375 BTU/HR [see section VII.A.2]

10. Total Heat Load = 1,238,275 BTU/HR

## 14. B. MAIN CONTROL ROOM COMPLEX SURFACE AREA AND VOLUME:

16. Main Control Room Complex Surface Area:

18. Slab thickness at elevation 140'-0" = 1'-0" [Ref. 5]

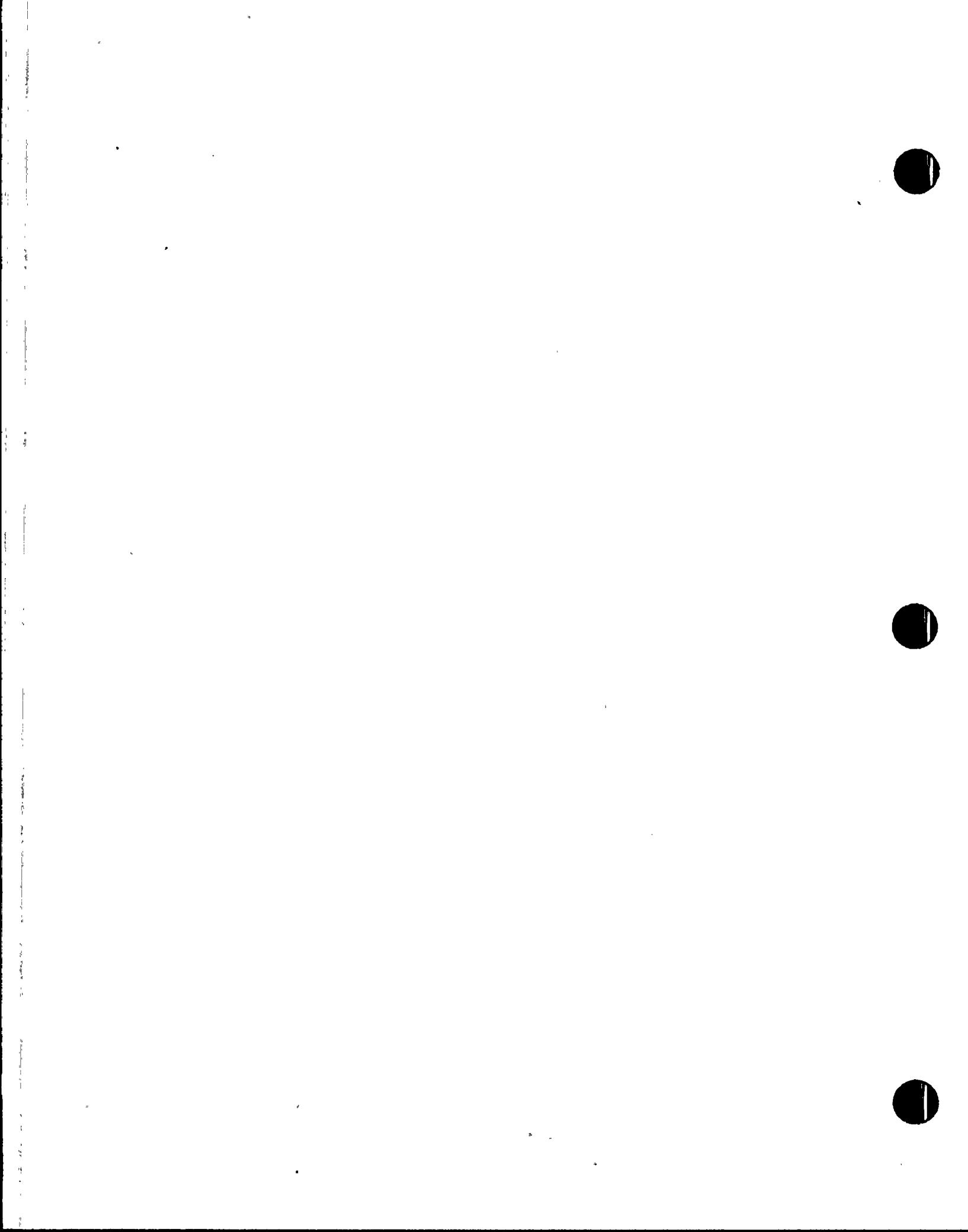
19. Slab thickness at elevation 160'-0" = 11" [Ref. 5]

20. Height of the control room = 160'-140'-(0'-11") = 19'-1"

21. Conservatively, use room height = 19'-0"

23. Thickness of the control room outside walls = 1'-9" [Ref. 5]

24. Thickness of the control room inside walls around cable  
25. shafts and HVAC chase = 1'-0" [Ref. 5]





PROJECT ANPP

## CALCULATION SHEET

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-253

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 24

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE	REV	INDICATOR
0	PSS	6-13-88	JM	6/13/88	1						

The following wall lengths are shown in Reference 5.

For 1'-9" thick wall

Length of East Side Wall =

$$(82' - 6") - (8' - 3") - (10' - 3") = 64' - 0"$$

Length of North Wall =

$$(110' - 6") - (13' - 3") - (20' - 3") = 77' - 0"$$

Length of West Wall =

$$(82' - 6") - (13' - 3") - (7' - 3") = 62' - 0"$$

Length of South Wall =

$$(110' - 6") - (12' - 9") - (24' - 3") = 73' - 6"$$

Total Wall length = (64' - 0") + (77' - 0") + (62' - 0")

$$+ (73' - 6") = 276' - 6"$$

Wall Surface Area (276' - 6") x (19' - 0") = 5253.5 ft<sup>2</sup>

Door area = (7' - 4 1/2") x (11' - 0") + (3' - 4 1/2")

$$\times (7' - 2 1/4") = 81.13 + 24.26 = 105.39 \text{ ft}^2$$

Net wall surface area = 5253.5 - 105.39 = 5148.11 ft<sup>2</sup>

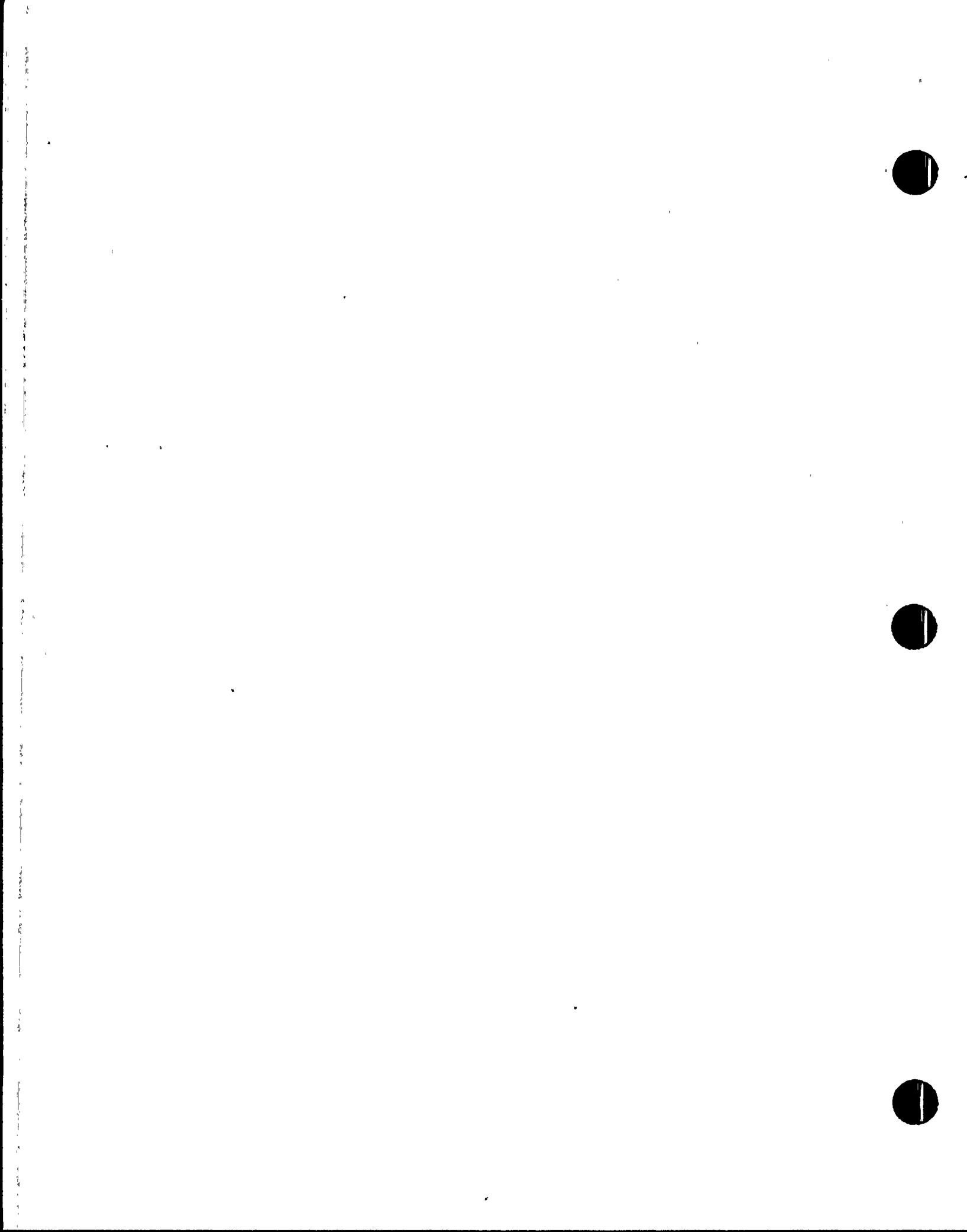
For 1'-0" Thick Walls:

The walls around the outside air chase are neglected because these walls act as a heat source during the initial period and act as a heat sink during the later period (Refer to Assumption 16).

The length of the walls around the cable shafts and HVAC chases in contact with the control room

$$= (13' - 3") + (10' - 3") + (13' - 3") + (20' - 3") + (12' - 9") + (7' - 3") + (17' - 3")$$

$$= 94' - 3"$$





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-253

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 25

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE	REV INDICATOR
0	PSS	6-13-88	Mun	6/13/88	1					▼
1					1					

Wall surface area =  $(94' - 3") \times (19' - 0")$   
= 1790.75 ft<sup>2</sup>

Less door area =  $(3' - 4 \frac{1}{2}") \times (7' - 2 \frac{1}{4"})$   
= 24.26 ft<sup>2</sup>

Net wall surface area = 1790.75 - 24.26 = 1766.49 ft<sup>2</sup>

Floor area =  $[(110' - 6") \times (82' - 6")] - [(13' - 3") \times (10' - 3")]$   
+  $(20' - 3") \times (13' - 3") + (12' - 9") \times (7' - 3")$   
+  $(24' - 3") \times (8' - 3")$   
= 9116.25 - 696.63 = 8419.62 ft<sup>2</sup>

Ceiling area (same as the floor area) = 8419.62 ft<sup>2</sup>

Therefore net surface area = 5148.11 + 1766.49 + 8419.62 +  
8419.62 = 23,753.84 ft<sup>2</sup>

Use net surface area = 23,754 ft<sup>2</sup>

As we are interested in short term results, all the heat transfer surfaces will be considered 1'-0" thick. The heat sink effect of the walls thicker than 1'-0" is significant only for the longer term. (Assumption 5).

Main Control Room Complex Volume

Floor area = 8419.62 ft<sup>2</sup> Height = 19' - 0"  
Volume =  $8419.62 \times 19 = 159,973 \text{ ft}^3$





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-253

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 26

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	5-13-88	Momz	6/13/88	1				

REV.  
INDICATOR  
▼1 C. Input Data:  
23 The following input data for the computer model is used:  
4

- 5 1. Initial room ambient temp. =  $75.5^{\circ}$  F [Assumption 1]
- 6 2. Initial outside ambient temp. =  $75.5^{\circ}$  F [Assumption 2]
- 7 3. Equipment and other net heat generated in the room =  
8 1,238,275 BTU/HR [See Section VII.A.3]
- 9 4. Net room surface area = 23,754 FT<sup>2</sup> [See Section VII.B]
- 10 5. Net room volume = 159,973 FT<sup>3</sup> [See Section VII.B]
- 11 6. Thickness of room enclosure = 1'-0" FT [Ref 5 and  
12 Assumption 5]
- 13 7. Density of room enclosure material = 144 LBS/FT<sup>3</sup>  
[Ref 12]
- 14 8. Thermal conductivity of room enclosure material =  
15 0.54 BTU/HR-FT- $^{\circ}$ F [Ref 12]
- 16 9. Specific heat of room enclosure material = 0.2 BTU/LB- $^{\circ}$ F  
17 [Ref 12]
- 18 10. One period of increment for calculation = 2 min  
19 [Assumption 10]
- 20 11. Imaginary thickness of first layer of concrete enclosure =  
21 0.01 FT [Assumption 11]
- 22 12. Multiplication factor of imaginary thickness of other  
23 layers = 1.41 [Assumption 11]

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## CALCULATION SHEET

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-25

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 27

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
10	PSS	6-13-88	Plasma	6/13/88	1				

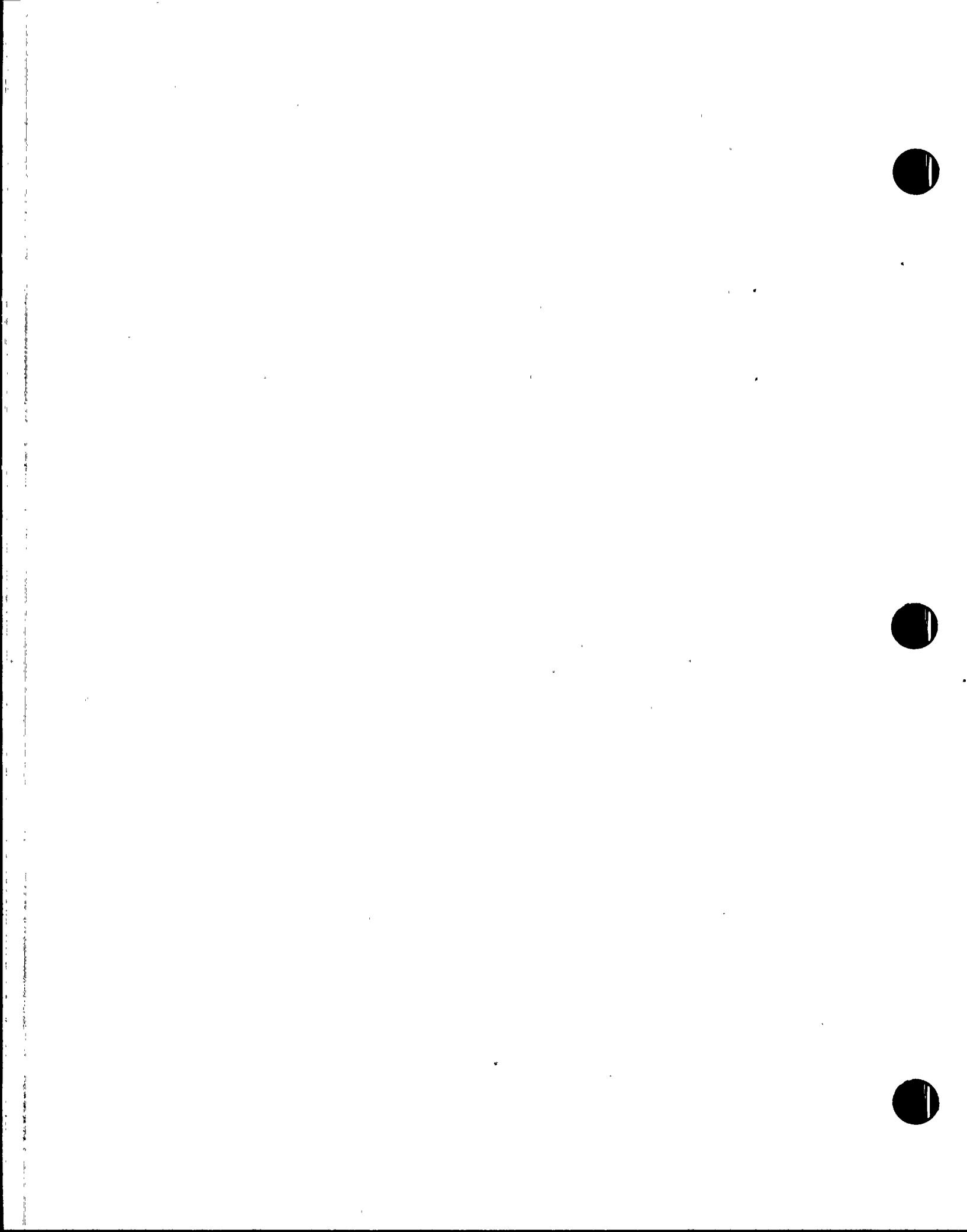
REV.  
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## Effect of AHUs on Control Room Complex Temperature

The two ESF AHU fans are 44% of the total heat load. To evaluate this sensitivity two additional computer runs are made for the cases where only one or none of the two ESF AHU fan motors are running. Only input.data item #3 in the above table is changed.

For one ESF AHU fan motor running, the heat load is  
 $1,238,275 - 273,950 = 964,325 \text{ BTU/HR.}$

For no ESF AHU fans running, the heat load is  
 $1,238,275 - 2 \times 273,950 = 690,375 \text{ BTU/HR.}$





# CALCULATION SHEET

PROJECT ANPPJOB NO. 18601 - 200CALC. NO. 13-MC-HJ-253SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOMSHEET NO. 28

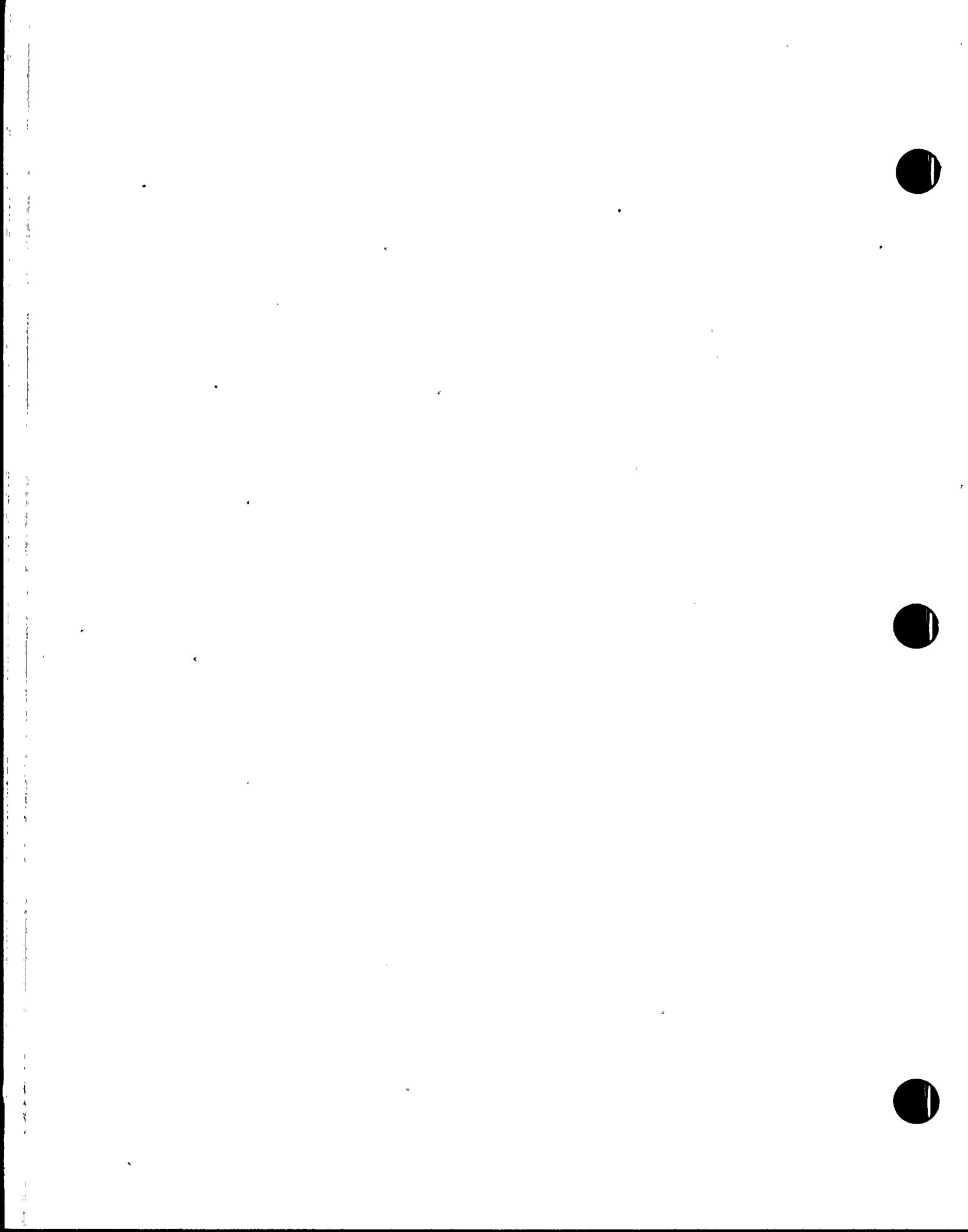
REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-88	Morris	6/13/88	0				
					0				

REV.  
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CHANGES

## 1      D. RESULTS:

2  
3      The transient room air temperature for the Main Control  
4      Room Complex obtained from the computer output, is shown in  
5      Figure 1. The transient temperature output, Appendix A,  
6      shows that with both essential AHU fans of Trains A & B,  
7      operating under conditions outlined in Section I, the  
8      calculated temperature in the pump room will be as shown  
9      below. For cases with one or no fans running, the  
10     transient temperature outputs, Appendices B and C, are also  
11     presented:  
12

Time Period	Appendix A Control Room w/2 ESF Fans	Appendix B Control Room w/1 ESF Fan	Appendix C Control Room w/No ESF Fans
	Temp. °F	Temp. °F	Temp. °F
0 min	75.5	75.5	75.5
2 min	86.07	83.79	81.49
4 min	93.65	89.78	85.86
6 min	99.00	94.06	89.02
12 min	107.43	100.95	94.23
36 min	115.21	107.38	99.19
60 min	118.50	110.00	101.12
84 min	121.05	112.03	102.60
96 min	122.18	112.92	103.25
2 hr.	124.21	114.52	104.42
4 hr	132.05	120.73	108.95
6 hr	138.02	125.45	112.38
8 hr	143.03	129.40	115.26
12 hr	151.46	136.06	120.10
24 hr	171.00	151.48	131.31





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-253

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 29

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-88	Morr	6/13/88	1				
					2				

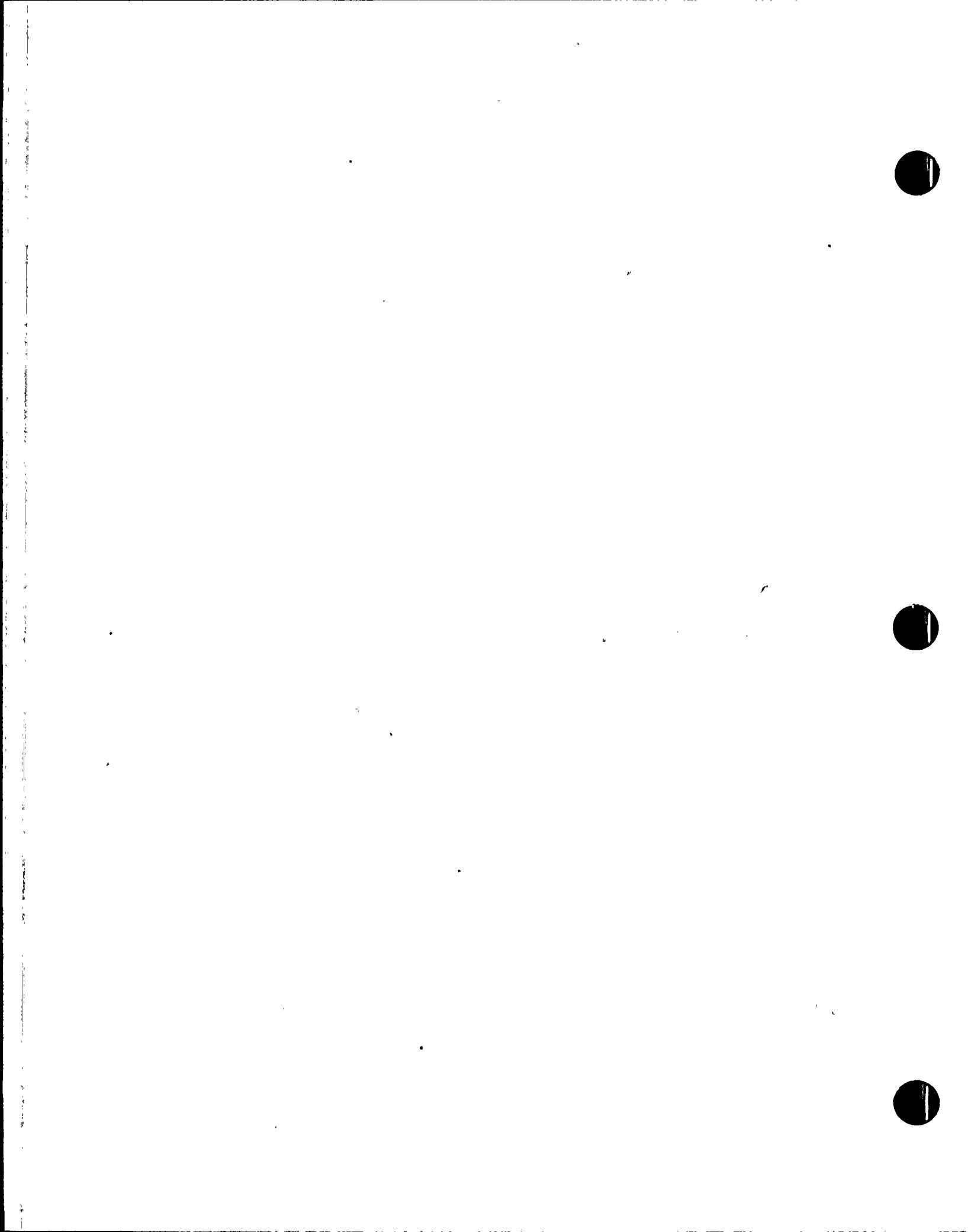
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## E. DISCUSSION of RESULTS:

As shown in Appendix A, the temperature inside the Main Control Room Complex is lower than the outside temperature of  $101.3^{\circ}\text{F}$  (Reference 16) for the first 6 to 8 minutes (for the base case of 2 fans running) and is higher after this time period. Therefore, a small percentage of the total heat will be added to the Control Room Complex by the outside makeup air during the first few minutes, while it will provide a very small amount of cooling after this time interval. The initial heat source effect is insignificant as shown in Section VII.F and is excluded.

Reviewing the above results, it is concluded that the temperature after the first 6 to 8 minutes (for the base case) is slightly overestimated due to the ventilation cooling effect of the two essential control room pressurization trains. Each of these two trains has a flow of 1000 cfm through the Main Control Room Complex (Ref. 14, page 2). The total ventilation cooling effect would only be on the order of 5% of the control room heat load when the control room temperature reached  $130^{\circ}\text{F}$ .

As can be readily seen by looking at the results of the cases with less than two ESF AHU fans operating, stopping one of the fans as soon as possible will dramatically decrease the rate of temperature rise in the control room complex. Stopping both fans may be undesirable, because "hot spots" could result in the various small rooms within the complex. Further evaluation would be required to assess this effect. Stopping both fans also is not consistent with the LOCA scenario described in section I.





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-253

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 30

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-88	Mowri	6/17/88	1				
1					2				

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## F. EFFECT OF OUTSIDE MAKEUP AIR

Outside makeup air flow for each ESF AHU train = 1000 cfm  
[Ref. 14, page 2]

Outside makeup air temperature = 101.3°F [Ref. 16]

Time required for the Main Control Room Complex to reach  
101.3°F, when both ESF AHU fans are operating = 6 to 8 minutes  
[Ref. Appendix A]

The makeup air flow will act as a heat source initially and  
later as a heat sink.

The average rate of heat added to the Main Control Room Complex  
due to both essential makeup trains before the control room  
temperature equals the outside makeup air temperature is

$$= 1.08 \times \text{cfm} \times \Delta T \quad [\text{Reference 1, Chapter 19, } \\ \text{Page 343, Equation 5}]$$

$$= 1.08 \times (2 \times 1000) \times 101.3 - 75.5 \text{ BTU/HR} \\ = 27,864 \text{ BTU/HR}$$

Therefore, heat load added to the Main Control Room Complex

$$= 27,864 \times 8 \text{ minutes} = 3715 \text{ BTU} \\ 60 \text{ minutes}$$





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-25

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 31

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-88	Monica	6/13/88	1				

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Specific heat of steel = 0.104 BTU/lb - °F [Ref. 12, table A-1]

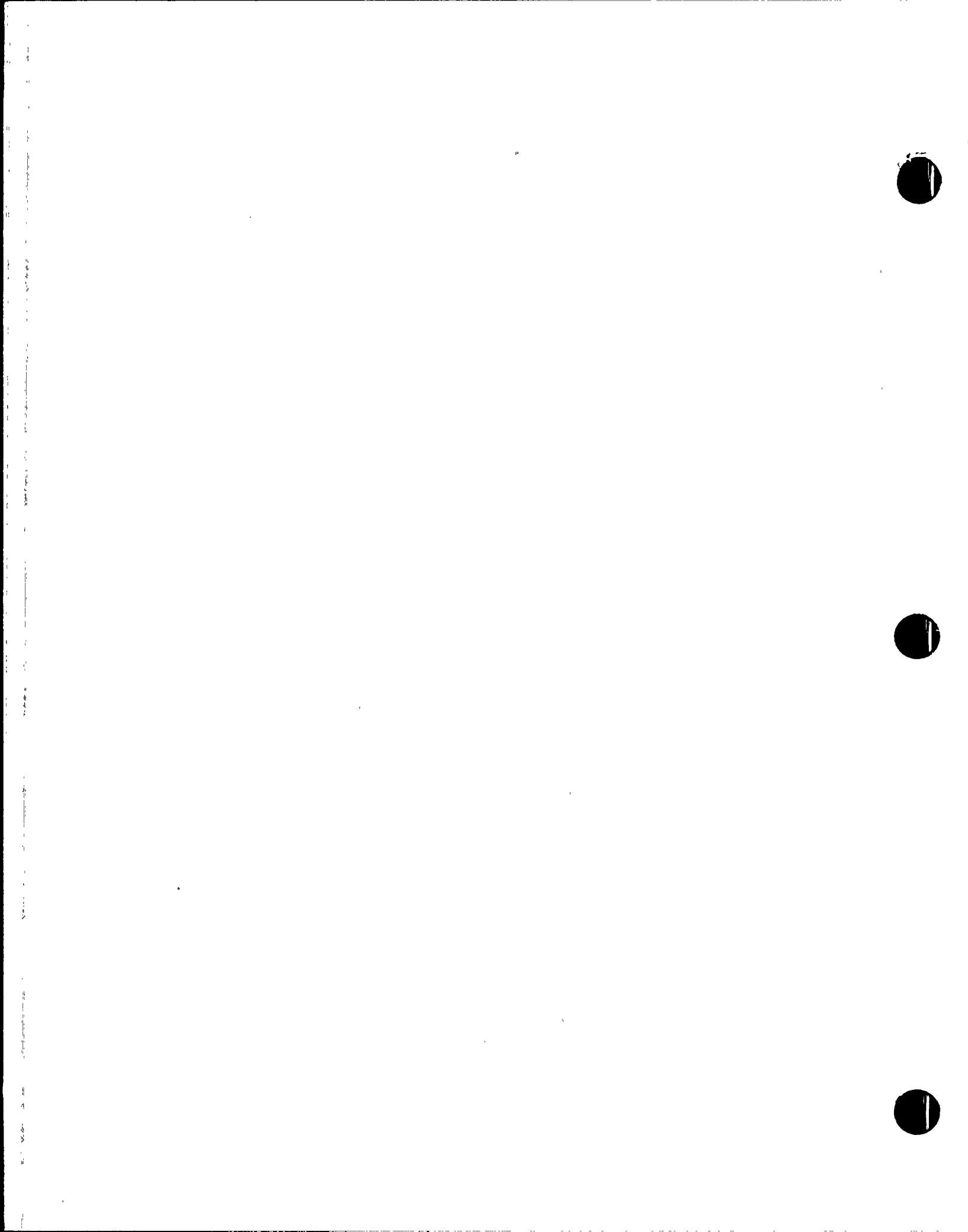
Weight of steel required to absorb the added heat load

$$= \frac{3715 \text{ BTU}}{0.104 \text{ BTU/lb} - °\text{F} \times (101.3 - 75.5) °\text{F}} = 1385 \text{ lb}$$

Similarly, for the cases of one and no ESF AHU fans running, the times at which the control room temperature equals the outside air temperature are approximately 15 minutes and 60 minutes respectively. The heat load added to the control room are 6,966 and 27,864 BTU requiring approximately 2,600, and 10,400 lb of steel, respectively, to absorb that heat.

By inspection, it is obvious that the Main Control Room has a large amount of steel consisting of panels, cabinets, beams, reinforcements, etc. that is many times more than the required weight of 10,400 lb.

Therefore, the heat source effect of the outside makeup air is insignificant and does not need to be included.





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-253

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 32

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
A	PSS	6-13-88	Denver	6/13/88	A				
A					A				

REV IND

In order to obtain slightly more accurate transient temperatures for the main control room, the Bechtel Computer Program NE-100 (COPPATTNA-7) may be used instead of ME-204. The computer program NE-100 has the following advantages over ME-204:

1. Heat Sink Model:

The metal items (e.g. cabinets, cable trays, structural steel and concrete) in the main control room can be modeled with much more complexity and accuracy rather than consideration as a lump sum mass.. This will lead to higher accuracy in predicting long term (e.g. 24 hours) room temperatures.

2. Effect of Outside Air:

The effect of the outside air in terms of the flow and temperature is accounted for in NE-100 and, therefore, represents a better temperature profile over the short term (e.g. 90 minutes).

3. Air Handling Units On and Off Simulation:

AHU performance is used as an input in the computer program NE-100. The temperature transient can be determined at any given time following the restoration of the chilled water through the AHU.



## **CALCULATION SHEET**

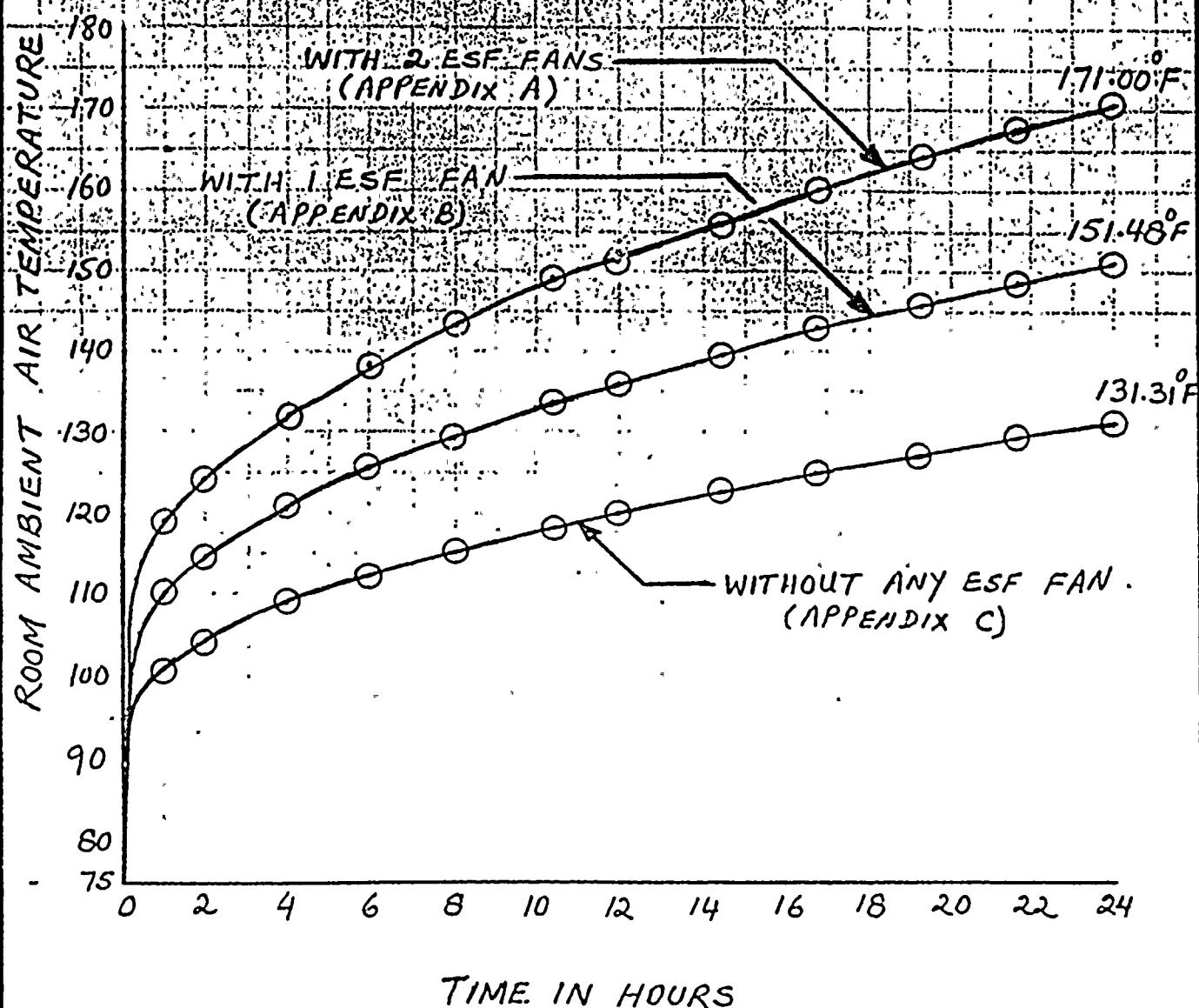
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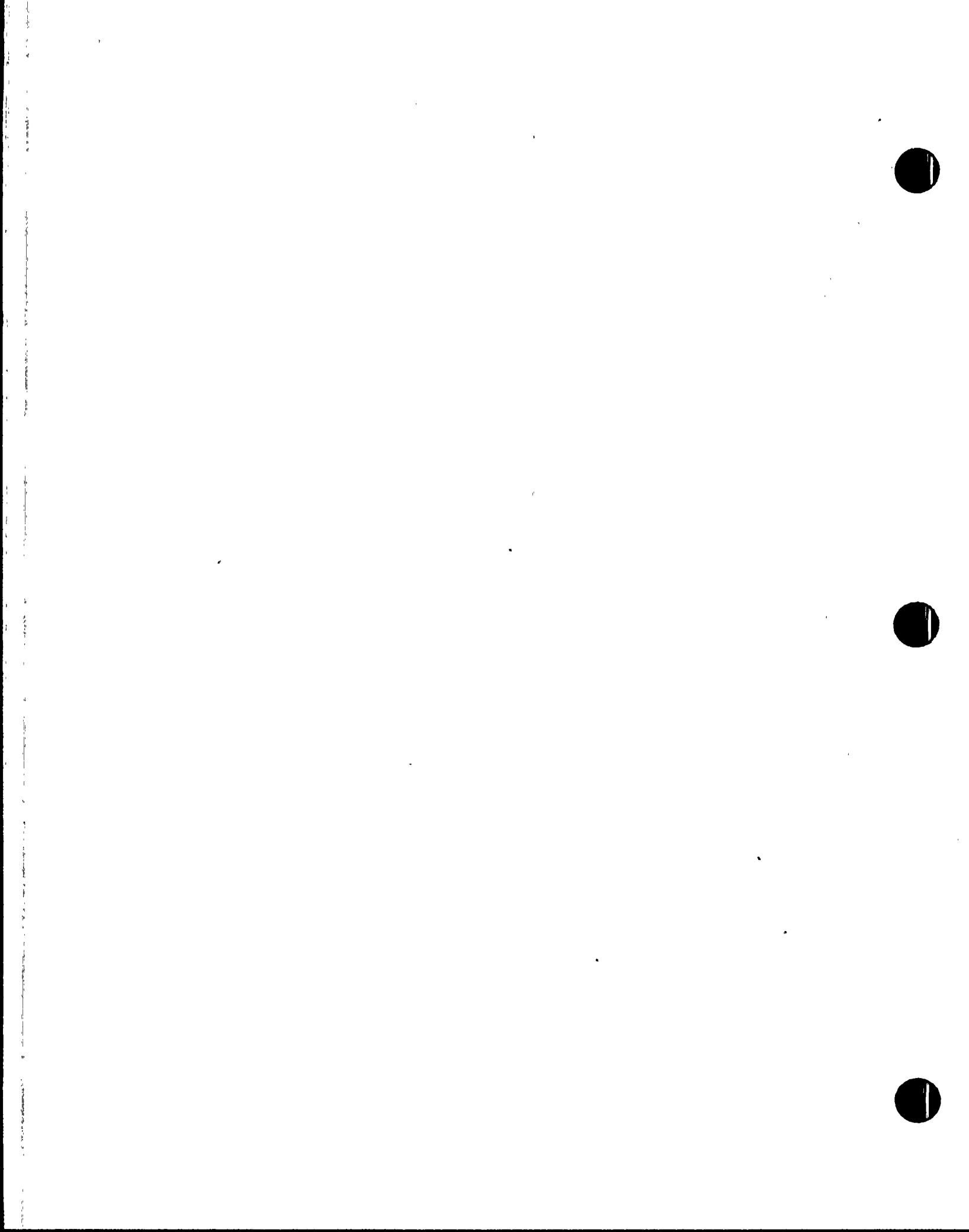
CALC. NO. 13-MC-HJ-25

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. - 33



## FIGURE 1





# CALCULATION SHEET

PROJECT ANPPJOB NO. 18601 - 200CALC. NO. 13-MC-HJ-25SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOMSHEET NO. 34

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-88	7/18/88	6/13/88	1				
1					1				

REV  
4 IND

## APPENDIX A

- INITIAL INTERNAL ROOM AMBIENT TEMPERATURE, DEG. F=?  
 >75.5  
 INITIAL ADJACENT ROOM AMBIENT TEMPERATURE, DEG. F=?  
 >75.5  
 EQUIPMENT HEAT GENERATED IN THE INTERNAL ROOM, BTU/HR=?  
 >1238275  
 NET INTERNAL ROOM SURFACE AREA, SQ. FT.=?  
 >23754  
 NET INTERNAL ROOM VOLUME, CU.FT.=?  
 >159973  
 THICKNESS OF ROOM ENCLOSURE, FT.=?  
 >1  
 DENSITY OF ROOM ENCLOSURE MATERIAL, LBS/CU.FT.=?  
 >144  
 THERMAL CONDUCTIVITY OF ROOM ENCLOSURE MATERIAL, BTU/HR-FT-F=?  
 >.54  
 SPECIFIC HEAT OF ROOM ENCLOSURE MATERIAL, BTU/LB-F=?  
 >.2  
 ONE PERIOD OF TIME INCREMENT FOR CALCULATION, MIN.=?  
 >2  
 IMAGINARY THICKNESS OF FIRST LAYER OF ROOM ENCLOSURE, FT.=?  
 >.01  
 MULTIPLICATION FACTOR OF IMAGINARY THICKNESS OF OTHER LAYERS=?  
 >1.41

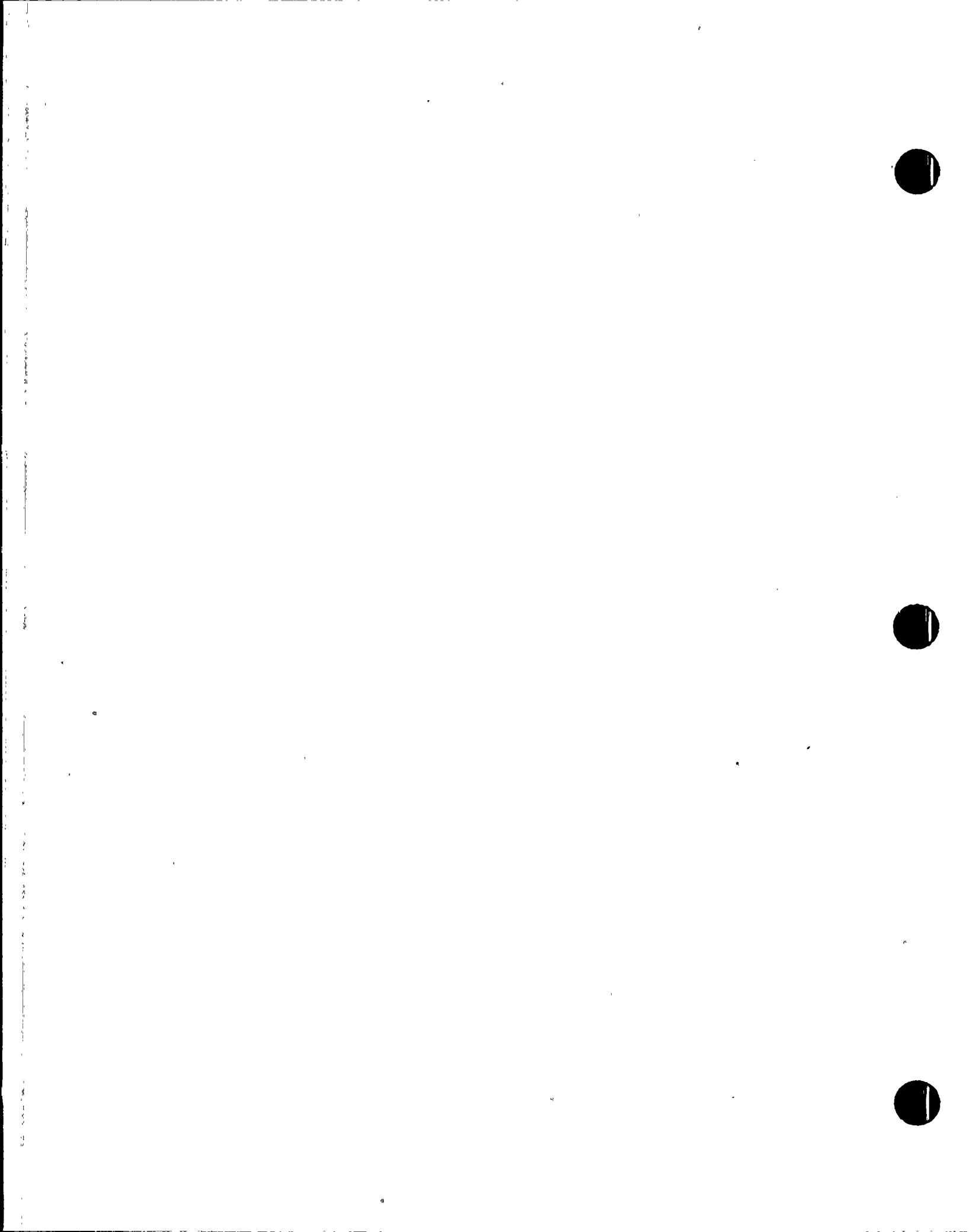
\*\*\*\*\*  
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 \*\*\*\*\*

M=NUMBER OF IMAGINARY LAYER= 11

DX1, DX2, DX3, -----DX(M)					
1.00000-02	1.41000-02	1.98810-02	2.80322-02	3.95254-02	5.57308-02
7.85805-02	.11080	.15623	.22028	.26685	

2 MIN	1 PERIOD QAT=.30146. QST= 11130. QOT= 0. HCl=1.4057 HCo=.0000				
TAF= 86.07					
T1, T2, T3, -----T(M+1)					
76.073	75.889	75.729	75.612	75.542	75.512
75.502	75.500	75.500	75.500	75.500	75.500

2 PERIOD QAT= 51331. QST= 31219. QOT= 0. HCl=1.5099 HCo=.0000					
TAF= 93.65					
T1, T2, T3, -----T(M+1)					
76.845	76.484	76.135	75.845	75.648	75.547
75.510	75.501	75.500	75.500	75.500	75.500





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-253

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 35

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE	REV INDICATOR
0	PSS	6-13-88	Donut	6/13/88	0					▼
1					1					▼

3 PERIOD QAT= 66090. QST= 57736. QOT= 0. HCI=1.5694 HCO=.0000  
TAF= 99.00

T1,T2,T3,-----T(M+1)  
77.659 77.158 76.639 76.169 75.817 75.612  
75.527 75.504 75.500 75.500 75.500 75.500

6 PERIOD QAT= 89025. QST= 158627. QOT= 0. HCI=1.6490 HCO=.9473  
TAF= 107.43

T1,T2,T3,-----T(M+1)  
79.875 79.138 78.287 77.394 76.583 75.984  
75.654 75.531 75.504 75.500 75.500 75.500

12 PERIOD QAT= 103455. QST= 391850. QOT= 0. HCI=1.6893 HCO=.9538  
TAF= 112.81

T1,T2,T3,-----T(M+1)  
83.033 82.172 81.091 79.817 78.450 77.182  
76.237 75.721 75.540 75.504 75.500 75.500

18 PERIOD QAT= 109848. QST= 633106. QOT= 2. HCI=1.7039 HCO=.9621  
TAF= 115.21

T1,T2,T3,-----T(M+1)  
85.286 84.396 83.246 81.827 80.195 78.516  
77.063 76.091 75.643 75.519 75.501 75.501

24 PERIOD QAT= 114581. QST= 876017. QOT= 9. HCI=1.7141 HCO=.9718  
TAF= 116.99

T1,T2,T3,-----T(M+1)  
87.117 86.214 85.031 83.535 81.751 79.808  
77.973 76.582 75.818 75.555 75.504 75.502

1HR 30 PERIOD QAT= 118574. QST= 1119654. QOT= 30. HCI=1.7226 HCO=.9823  
TAF= 118.50

T1,T2,T3,-----T(M+1)  
88.702 87.792 86.587 85.042 83.156 81.027  
78.900 77.145 76.058 75.616 75.510 75.507

36 PERIOD QAT= 122103. QST= 1363727. QOT= 80. HCI=1.7301 HCO=.9930  
TAF= 119.84

T1,T2,T3,-----T(M+1)  
90.121 89.205 87.985 86.405 84.446 82.176  
79.818 77.750 76.349 75.704 75.521 75.514

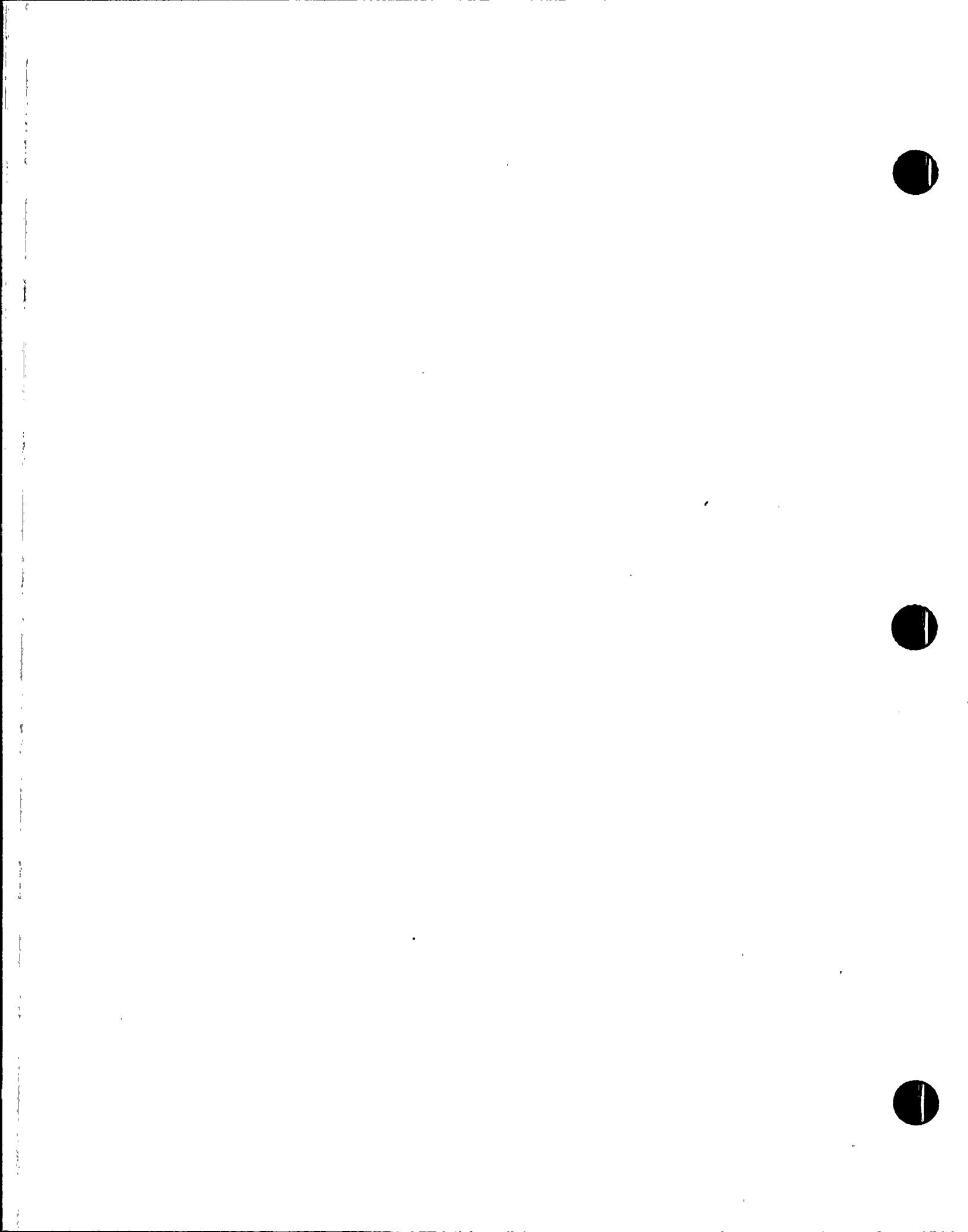
42 PERIOD QAT= 125301. QST= 1608083. QOT= 177. HCI=1.7370 HCO=1.0039  
TAF= 121.05

T1,T2,T3,-----T(M+1)  
91.417 90.497 89.266 87.659 85.643 83.263  
80.716 78.377 76.680 75.817 75.538 75.526

48 PERIOD QAT= 128247. QST= 1852622. QOT= 345. HCI=1.7433 HCO=1.0147  
TAF= 122.18

T1,T2,T3,-----T(M+1)  
92.617 91.694 90.454 88.826 86.764 84.295  
81.590 79.015 77.043 75.956 75.564 75.542

2HR 60 PERIOD QAT= 133569. QST= 2341939. QOT= 1007. HCI=1.7548 HCO=1.0358  
TAF= 124.21





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-25

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 36

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
A	PSS	5-13-88	Morris	5/13/88	A				
					A				

REV.

2 T1,T2,T3, -----T(M+1)  
 3 94.801 93.873 92.619 90.959 88.828 86.221  
 3 83.266 80.298 77.832 76.295 75.641 75.593

5 72 PERIOD QAT= 138328. QST= 2831166. QOT= 2323. HCI=1.7652 HCO=1.0559  
 TAF= 126.03

6 T1,T2,T3, -----T(M+1)  
 7 96.766 95.835 94.571 92.888 90.705 87.995  
 7 84.849 81.568 78.675 76.702 75.757 75.669

8 84 PERIOD QAT= 142670. QST= 3319877. QOT= 4572. HCI=1.7747 HCO=1.0749  
 TAF= 127.69

10 T1,T2,T3, -----T(M+1)  
 10 98.568 97.634 96.363 94.661 92.440 89.650  
 11 86.351 82.811 79.548 77.161 75.914 75.770

12 96 PERIOD QAT= 146687. QST= 3807692. QOT= 8043. HCI=1.7836 HCO=1.0928  
 TAF= 129.24

13 T1,T2,T3, -----T(M+1)  
 13 100.24 99.305 98.028 96.312 94.059 91.204  
 13 87.781 84.024 80.435 77.661 76.110 75.896

16 108 PERIOD QAT= 150441. QST= 4294262. QOT= 13019. HCI=1.7919 HCO=1.1097  
 TAF= 130.69

17 T1,T2,T3, -----T(M+1)  
 17 101.81 100.87 99.591 97.863 95.584 92.675  
 18 89.147 85.204 81.326 78.193 76.344 76.045

19 4HR 120 PERIOD QAT= 153976. QST= 4779273. QOT= 19773. HCI=1.7999 HCO=1.1256  
 TAF= 132.05

21 T1,T2,T3, -----T(M+1)  
 21 103.29 102.35 101.07 99.330 97.029 94.074  
 22 90.456 86.351 82.215 78.749 76.612 76.215

23 132 PERIOD QAT= 157325. QST= 5262439. QOT= 28557. HCI=1.8075 HCO=1.1406  
 TAF= 133.35

25 T1,T2,T3, -----T(M+1)  
 25 104.70 103.76 102.47 100.73 98.405 95.410  
 26 91.715 87.466 83.099 79.326 76.912 76.403

27 144 PERIOD QAT= 160514. QST= 5743502. QOT= 39605. HCI=1.8147 HCO=1.1547  
 TAF= 134.59

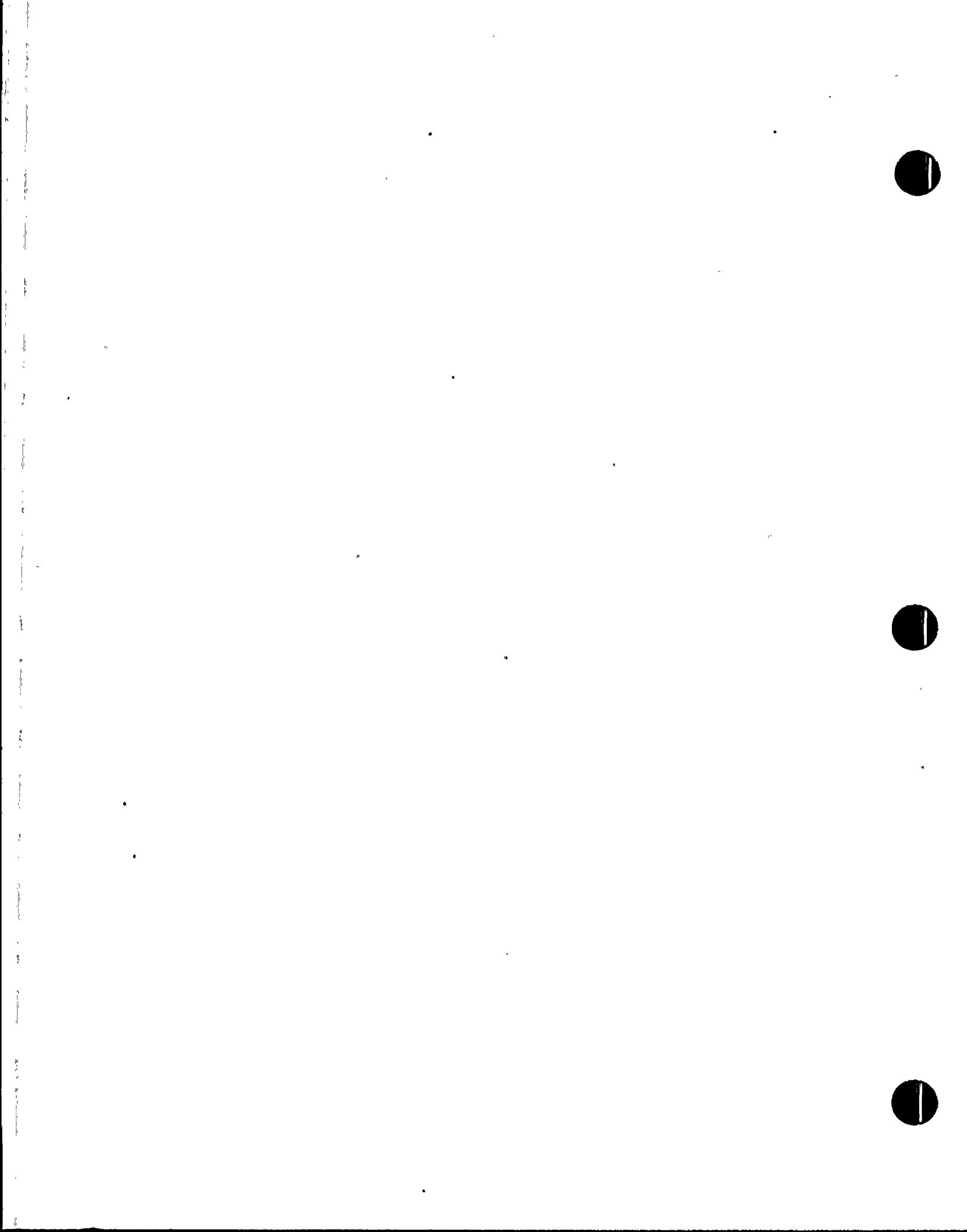
28 T1,T2,T3, -----T(M+1)  
 28 106.05 105.11 103.81 102.06 99.721 96.692  
 29 92.927 88.552 83.976 79.920 77.239 76.607

30 156 PERIOD QAT= 163563. QST= 6222226. QOT= 53131. HCI=1.8217 HCO=1.1681  
 TAF= 135.78

32 T1,T2,T3, -----T(M+1)  
 32 107.34 106.39 105.10 103.34 100.98 97.924  
 33 94.099 89.610 84.844 80.526 77.592 76.826

34 168 PERIOD QAT= 166488. QST= 6698404. QOT= 69327. HCI=1.8284 HCO=1.1808  
 TAF= 136.92

35 T1,T2,T3, -----T(M+1)  
 35 108.57 107.63 106.33 104.57 102.20 99.113  
 36 95.233 90.642 85.704 81.144 77.966 77.057





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-25

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO.

37

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-88	Thomas	6/13/88	1				

REV.  
4-INDI

3 6 HR 180 PERIOD QAT= 169303. QST= 7171850. QOT= 88365. HCI=1.8350 HCO=1.1929  
 4 TAF= 138.02  
 5 T1,T2,T3, -----T(M+1)  
 6 109.77 108.82 107.52 105.75 103.38 100.26  
 7 96.333 91.649 86.554 81.770 78.359 77.299

7 192 PERIOD QAT= 172017. QST= 7642399. QOT= 110400. HCI=1.8413 HCO=1.2043  
 8 TAF= 139.08  
 9 T1,T2,T3, -----T(M+1)  
 10 110.92 109.98 108.68 106.90 104.51 101.38  
 11 97.402 92.634 87.395 82.403 78.769 77.549

10 204 PERIOD QAT= 174642. QST= 8109906. QOT= 135567. HCI=1.8474 HCO=1.2153  
 11 TAF= 140.11  
 12 T1,T2,T3, -----T(M+1)  
 13 112.04 111.09 109.79 108.01 105.61 102.46  
 14 98.443 93.598 88.227 83.042 79.193 77.807

15 216 PERIOD QAT= 177186. QST= 8574241. QOT= 163986. HCI=1.8534 HCO=1.2257  
 16 TAF= 141.11  
 17 T1,T2,T3, -----T(M+1)  
 18 113.13 112.18 110.87 109.09 106.68 103.51  
 19 99.458 94.543 89.049 83.684 79.628 78.071

22 228 PERIOD QAT= 179655. QST= 9035290. QOT= 195762. HCI=1.8593 HCO=1.2356  
 23 TAF= 142.08  
 24 T1,T2,T3, -----T(M+1)  
 25 114.18 113.23 111.93 110.14 107.72 104.53  
 26 100.45 95.469 89.864 84.330 80.074 78.340

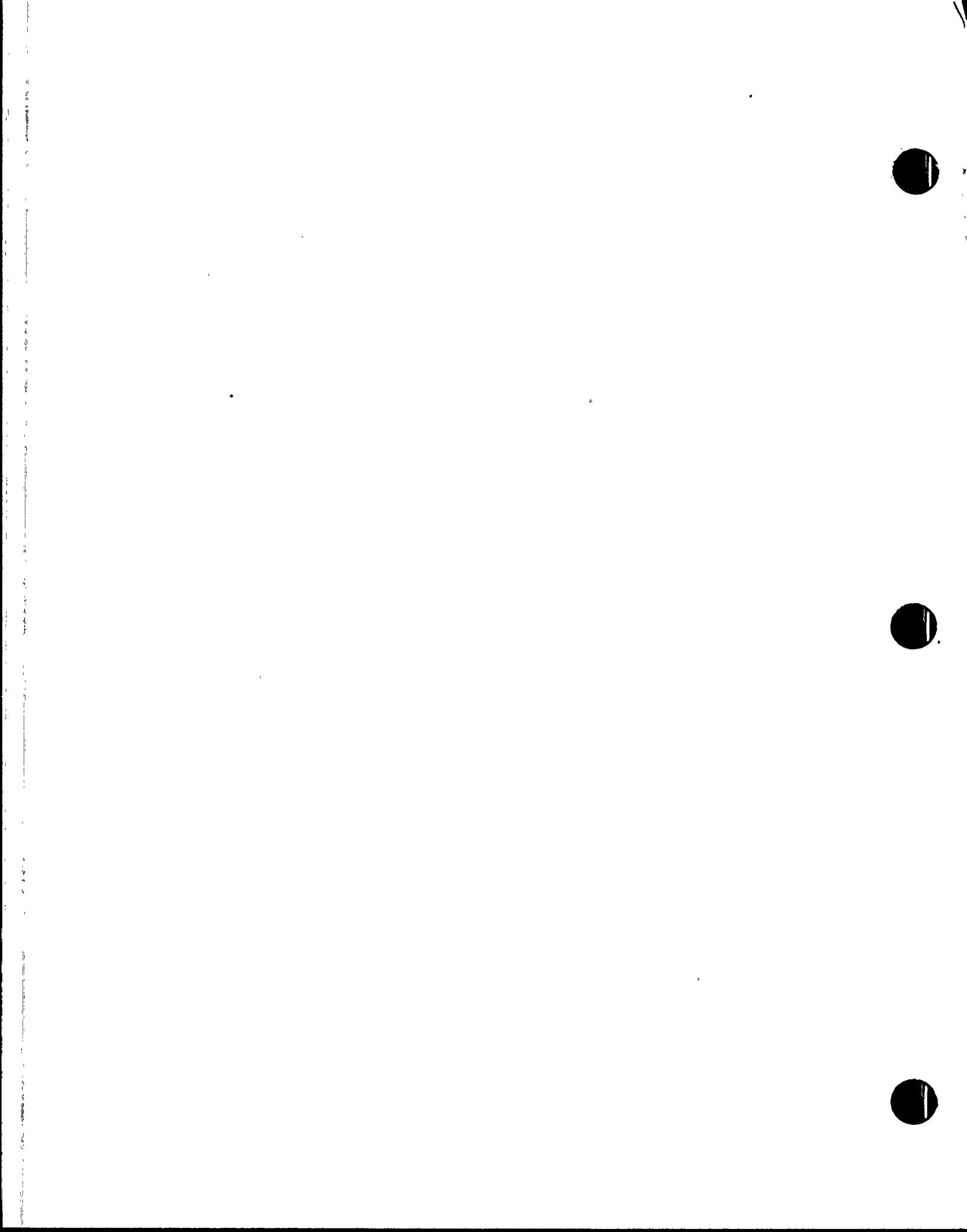
21 8 HR 240 PERIOD QAT= 182056. QST= 9492960. QOT= 230985. HCI=1.8650 HCO=1.2451  
 22 TAF= 143.03  
 23 T1,T2,T3, -----T(M+1)  
 24 115.21 114.26 112.95 111.16 108.74 105.53  
 25 101.42 96.379 90.669 84.977 80.528 78.613

25 264 PERIOD QAT= 186678. QST= 10397839. QOT= 312073. HCI=1.8761 HCO=1.2630  
 26 TAF= 144.85  
 27 T1,T2,T3, -----T(M+1)  
 28 117.19 116.24 114.93 113.13 110.70 107.47  
 29 103.30 98.151 92.255 86.274 81.455 79.167

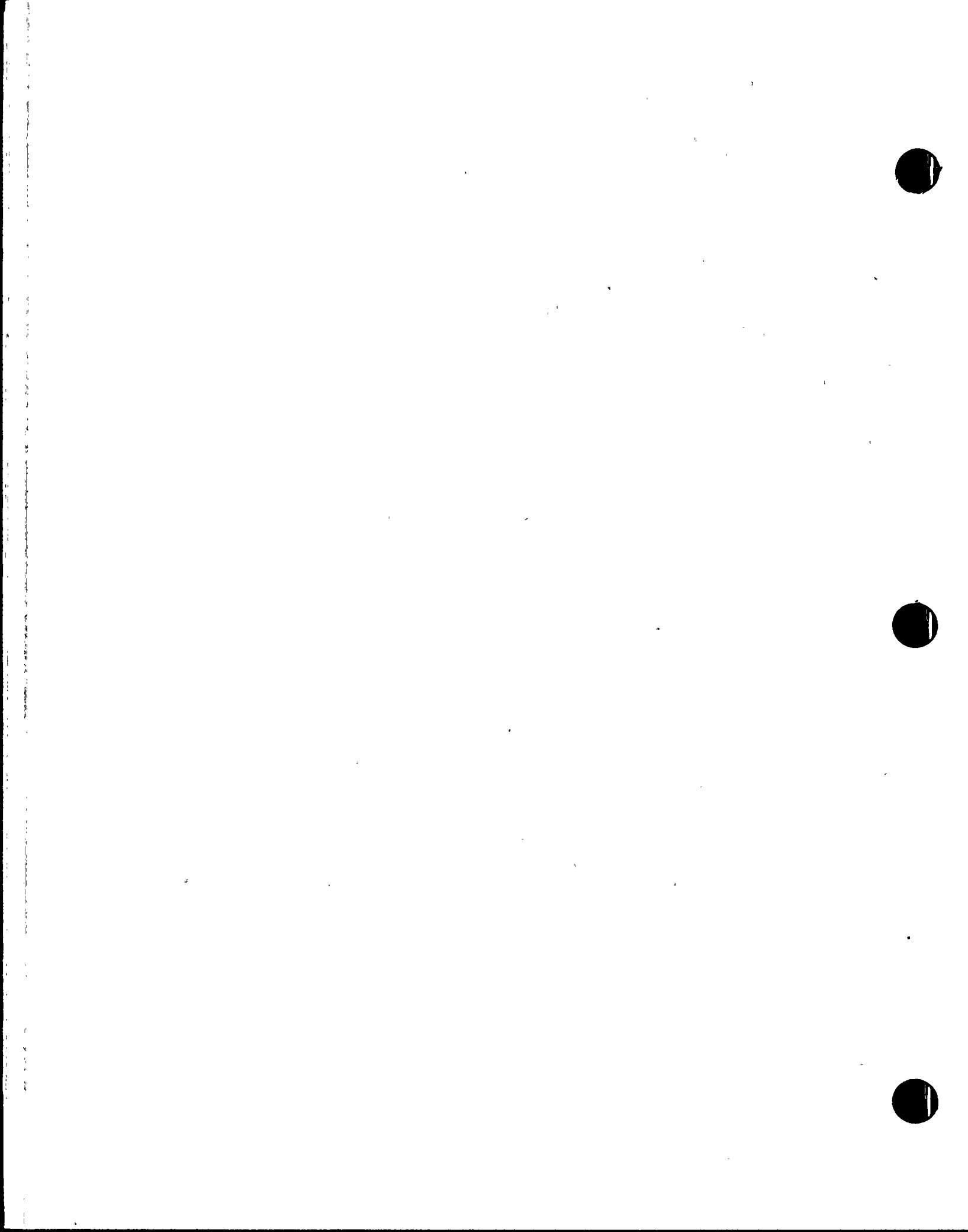
29 288 PERIOD QAT= 191087. QST= 11288342. QOT= 407749. HCI=1.8868 HCO=1.2795  
 30 TAF= 146.60  
 31 T1,T2,T3, -----T(M+1)  
 32 119.09 118.14 116.83 115.02 112.58 109.32  
 33 105.10 99.868 93.809 87.570 82.400 79.727

33 312 PERIOD QAT= 195315. QST= 12164094. QOT= 518358. HCI=1.8971 HCO=1.2947  
 34 TAF= 148.27  
 35 T1,T2,T3, -----T(M+1)  
 36 120.91 119.96 118.65 116.84 114.39 111.11  
 37 106.85 101.54. 95.332 88.858 83.354 80.290

36 376 DEPTOQ QAT= 199785. QST= 13024844. QOT= 644126. HCI=1.9072 HCO=1.3089









# CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-2

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO.

39

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-68	Homer	6/13/68	1			-	

## APPENDIX B

- 4 INITIAL INTERNAL ROOM AMBIENT TEMPERATURE, DEG. F=?  
 5 >75.5  
 6 INITIAL ADJACENT ROOM AMBIENT TEMPERATURE, DEG. F=?  
 7 >75.5  
 8 EQUIPMENT HEAT GENERATED IN THE INTERNAL ROOM, BTU/HR=?  
 9 >964325  
 10 NET INTERNAL ROOM SURFACE AREA, SQ. FT.=?  
 11 >23754  
 12 NET INTERNAL ROOM VOLUME, CU.FT.=?  
 13 >159973  
 14 THICKNESS OF ROOM ENCLOSURE, FT.=?  
 15 >1  
 16 DENSITY OF ROOM ENCLOSURE MATERIAL, LBS/CU.FT.=?  
 17 >144  
 18 THERMAL CONDUCTIVITY OF ROOM ENCLOSURE MATERIAL, BTU/HR-FT-F=?  
 19 >.54  
 20 SPECIFIC HEAT OF ROOM ENCLOSURE MATERIAL, BTU/LB-F=?  
 21 >.2  
 22 ONE PERIOD OF TIME INCREMENT FOR CALCULATION, MIN.=?  
 23 >2  
 24 IMAGINARY THICKNESS OF FIRST LAYER OF ROOM ENCLOSURE, FT.=?  
 25 >.01  
 26 MULTIPLICATION FACTOR OF IMAGINARY THICKNESS OF OTHER LAYERS=?  
 27 >1.41

\*\*\*\*  
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 \*\*\*\*

25 M=NUMBER OF IMAGINARY LAYER= 11

26 DX1,DX2,DX3,-----DX(M)					
1.00000-02	1.41000-02	1.98810-02	2.80322-02	3.95254-02	5.57308-02
7.85805-02	.11080	.15623	.22028	.26685	

29 2 MIN 1 PERIOD QAT= 23648. QST= 8495. QOT= 0. HCl=1.3656 HCo=.0000					
TAF= 83.79					
30 T1,T2,T3,-----T(M+1)					
75.937	75.797	75.675	75.585	75.532	75.509
75.502	75.500	75.500	75.500	75.500	75.500

33 2 PERIOD QAT= 40472. QST= 23816. QOT= 0. HCl=1.4593 HCo=.0000					
TAF= 89.78					
34 T1,T2,T3,-----T(M+1)					
76.526	76.251	75.984	75.763	75.613	75.536
75.507	75.501	75.500	75.500	75.500	75.500





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-2E

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO.

40

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-88	Hornes	6/13/88	A				
					A				

REV.

3 PERIOD QAT= 52355. QST= 44076. QOT= 0. HCl=1.5129 HCo=.0000  
 TAF= 94.06

T1, T2, T3, -----T(M+1)  
 77.149 76.766 76.370 76.011 75.742 75.585  
 75.520 75.503 75.500 75.500 75.500 75.500

6 PERIOD QAT= 71261. QST= 121600. QOT= 0. HCl=1.5848 HCo=.9468  
 TAF= 100.95

T1, T2, T3, -----T(M+1)  
 78.859 78.292 77.637 76.951 76.329 75.871  
 75.618 75.524 75.503 75.500 75.500 75.500

12 PERIOD QAT= 83423. QST= 302301. QOT= 0. HCl=1.6211 HCo=.9530  
 TAF= 105.43

T1, T2, T3, -----T(M+1)  
 81.327 80.658 79.820 78.832 77.775 76.795  
 76.066 75.670 75.531 75.503 75.500 75.500

18 PERIOD QAT= 88687. QST= 489897. QOT= 1. HCl=1.6331 HCo=.9607  
 TAF= 107.38

T1, T2, T3, -----T(M+1)  
 83.089 82.397 81.503 80.400 79.133 77.832  
 76.707 75.956 75.610 75.515 75.501 75.500

24 PERIOD QAT= 92522. QST= 678918. QOT= 7. HCl=1.6412 HCo=.9696  
 TAF= 108.80

T1, T2, T3, -----T(M+1)  
 84.520 83.817 82.896 81.733 80.346 78.837  
 77.414 76.336 75.746 75.542 75.503 75.502

1 HR 30 PERIOD QAT= 95742. QST= 868542. QOT= 23. HCl=1.6479 HCo=.9791

TAF= 110.00  
 T1, T2, T3, -----T(M+1)  
 85.757 85.048 84.111 82.909 81.442 79.787  
 78.136 76.774 75.931 75.589 75.507 75.505

36 PERIOD QAT= 98585. QST= 1058523. QOT= 61. HCl=1.6538 HCo=.9890  
 TAF= 111.07

T1, T2, T3, -----T(M+1)  
 86.863 86.150 85.201 83.972 82.447 80.683  
 78.850 77.244 76.157 75.657 75.516 75.511

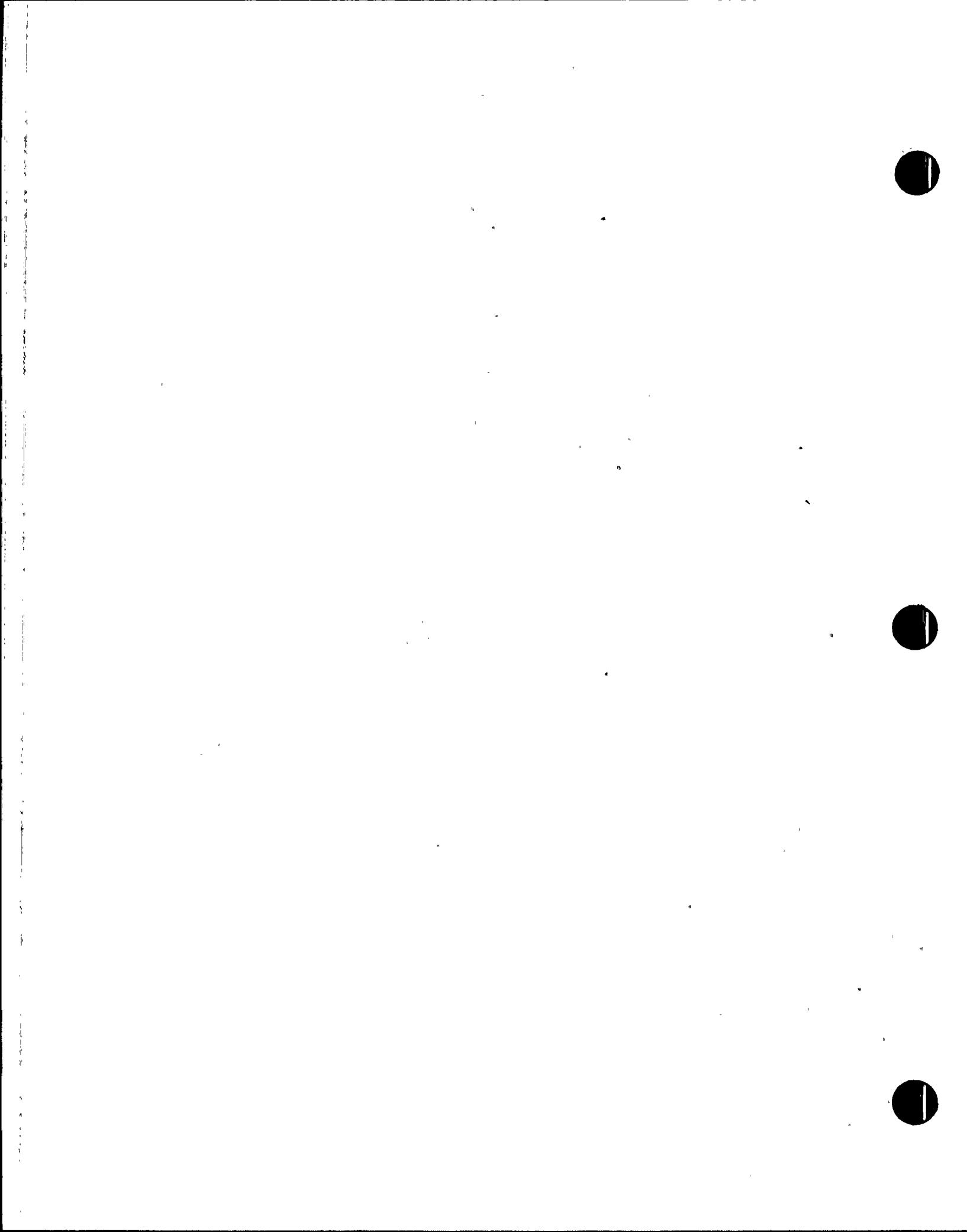
42 PERIOD QAT= 101160. QST= 1248735. QOT= 136. HCl=1.6591 HCo=.9990  
 TAF= 112.03

T1, T2, T3, -----T(M+1)  
 87.874 87.158 86.199 84.949 83.380 81.529  
 79.549 77.732 76.415 75.746 75.530 75.520

48 PERIOD QAT= 103530. QST= 1439096. QOT= 265. HCl=1.6640 HCo=1.0090  
 TAF= 112.92

T1, T2, T3, -----T(M+1)  
 88.810 88.091 87.126 85.859 84.254 82.333  
 80.230 78.228 76.696 75.853 75.549 75.533

2 HR 60 PERIOD QAT= 107814. QST= 1820025. QOT= 776. HCl=1.6729 HCo=1.0284  
 TAF= 114.52





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-2<sup>c</sup>

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO.

41

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
10	PSS	6-13-88	Douglas	6/13/88	10				

1  
2 T1, T2, T3, -----T(M+1)  
3 90.512 89.789 88.813 87.521 85.862 83.834  
4 81.535 79.227 77.310 76.116 75.609 75.572

5 72 PERIOD QAT= 111645. QST= 2200903. QOT= 1789. HCI=1.6809 HCO=1.0469  
6 TAF= 115.97  
7 T1, T2, T3, -----T(M+1)  
8 92.043 91.318 90.334 89.024 87.325 85.216  
9 82.768 80.215 77.966 76.433 75.699 75.631

10 84 PERIOD QAT= 115140. QST= 2581397. QOT= 3522. HCI=1.6882 HCO=1.0644  
11 TAF= 117.28  
12 T1, T2, T3, -----T(M+1)  
13 93.447 92.720 91.731 90.406 88.677 86.505  
14 83.938 81.184 78.646 76.790 75.821 75.710

15 96 PERIOD QAT= 118374. QST= 2961211. QOT= 6196. HCI=1.6950 HCO=1.0808  
16 TAF= 118.51  
17 T1, T2, T3, -----T(M+1)  
18 94.751 94.022 93.028 91.692 89.938 87.716  
19 85.052 82.128 79.336 77.179 75.973 75.809

20 108 PERIOD QAT= 121397. QST= 3340078. QOT= 10028. HCI=1.7014 HCO=1.0963  
21 TAF= 119.65  
22 T1, T2, T3, -----T(M+1)  
23 95.974 95.244 94.246 92.901 91.126 88.861  
24 86.116 83.047 80.030 77.593 76.155 75.925

25 4 HR 120 PERIOD QAT= 124245. QST= 3717751. QOT= 15228. HCI=1.7075 HCO=1.1109  
26 TAF= 120.73  
27 T1, T2, T3, -----T(M+1)  
28 97.129 96.398 95.396 94.043 92.252 89.951  
29 87.135 83.940 80.722 78.026 76.364 76.058

30 132 PERIOD QAT= 126944. QST= 4094013. QOT= 21991. HCI=1.7133 HCO=1.1247  
31 TAF= 121.76  
32 T1, T2, T3, -----T(M+1)  
33 98.227 97.494 96.490 95.130 93.324 90.992  
34 88.115 84.809 81.410 78.475 76.597 76.205

35 144 PERIOD QAT= 129514. QST= 4468658. QOT= 30494. HCI=1.7188 HCO=1.1377  
36 TAF= 122.74  
37 T1, T2, T3, -----T(M+1)  
38 99.274 98.541 97.534 96.169 94.349 91.990  
39 89.060 85.655 82.093 78.937 76.851 76.365

40 156 PERIOD QAT= 131972. QST= 4841512. QOT= 40902. HCI=1.7241 HCO=1.1500  
41 TAF= 123.68  
42 T1, T2, T3, -----T(M+1)  
43 100.28 99.544 98.535 97.164 95.333 92.950  
44 89.972 86.478 82.769 79.409 77.126 76.536

45 168 PERIOD QAT= 134331. QST= 5212414. QOT= 53363. HCI=1.7292 HCO=1.1617  
46 TAF= 124.58  
47 T1, T2, T3, -----T(M+1)  
48 101.24 100.51 99.497 98.122 96.281 93.876  
49 90.856 87.282 83.438 79.890 77.417 76.718





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-25

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 42

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-88	Morris	6/13/88	1				

REV.

1  
2  
3 6 HR 180 PERIOD QAT= 136601. QST= 5581218. QOT= 68007. HCI=1.7342 HCO=1.1727  
4 TAF= 125.45  
5 T1, T2, T3, -----T(M+1)  
6 102.17 101.44 100.42 99.045 97.195 94.772  
7 91.712 88.066 84.100 80.377 77.723 76.907

8 192 PERIOD QAT= 138791. QST= 5947801. QOT= 84954. HCI=1.7390 HCO=1.1832  
9 TAF= 126.29  
10 T1, T2, T3, -----T(M+1)  
11 103.07 102.34 101.32 99.938 98.080 95.639  
12 92.545 88.834 84.755 80.870 78.042 77.104

13 204 PERIOD QAT= 140909. QST= 6312049. QOT= 104308. HCI=1.7437 HCO=1.1932  
14 TAF= 127.10  
15 T1, T2, T3, -----T(M+1)  
16 103.94 103.21 102.19 100.80 98.938 96.481  
17 93.356 89.584 85.403 81.367 78.372 77.307

18 216 PERIOD QAT= 142961. QST= 6673865. QOT= 126159. HCI=1.7482 HCO=1.2028  
19 TAF= 127.89  
20 T1, T2, T3, -----T(M+1)  
21 104.79 104.05 103.03 101.64 99.771 97.300  
22 94.146 90.320 86.043 81.867 78.712 77.514

23 228 PERIOD QAT= 144955. QST= 7033162. QOT= 150589. HCI=1.7526 HCO=1.2119  
24 TAF= 128.65  
25 T1, T2, T3, -----T(M+1)  
26 105.61 104.87 103.85 102.46 100.58 98.099  
27 94.918 91.041 86.677 82.370 79.059 77.726

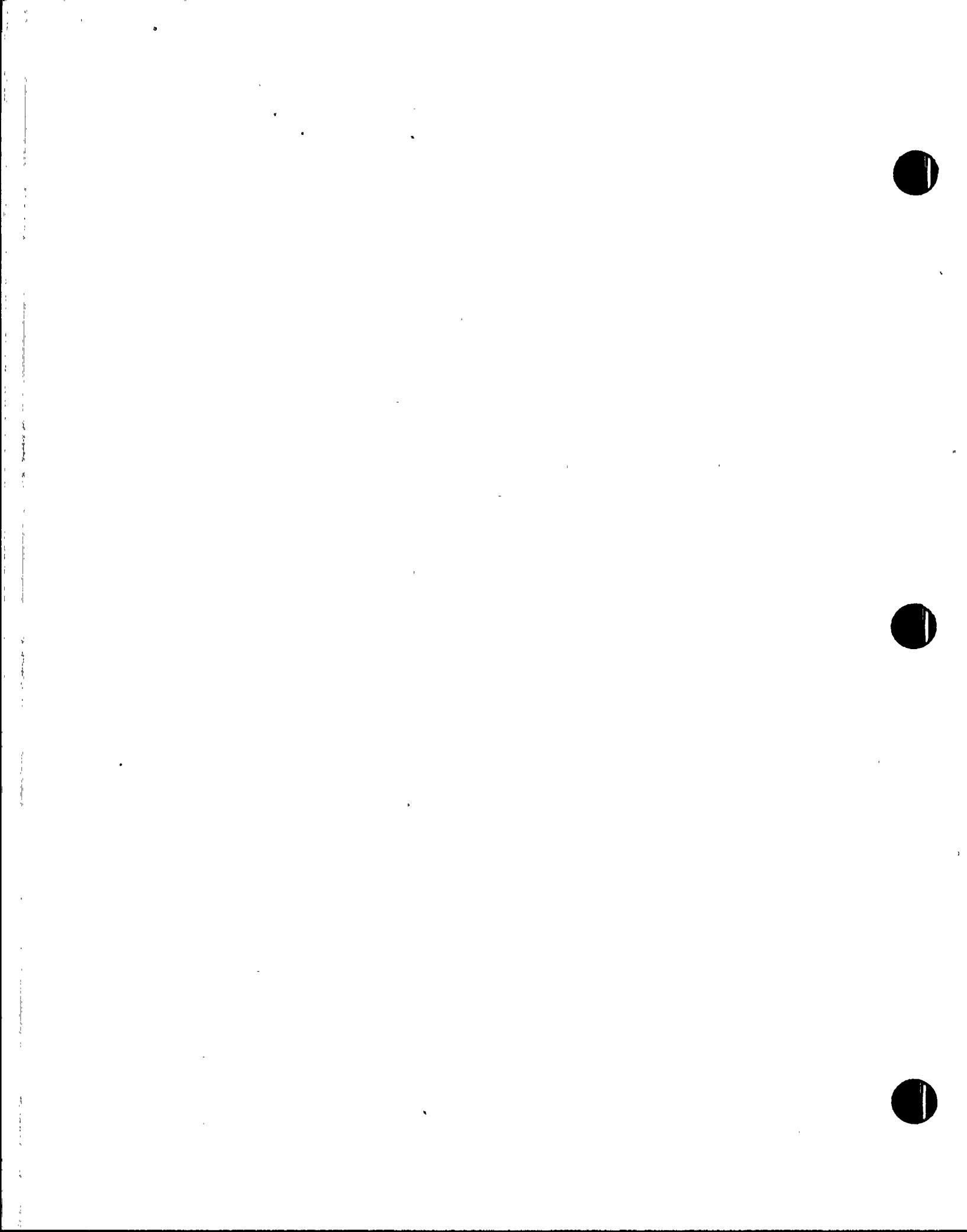
28 8 HR 240 PERIOD QAT= 146893. QST= 7389865. QOT= 177666. HCI=1.7570 HCO=1.2206  
29 TAF= 129.40  
30 T1, T2, T3, -----T(M+1)  
31 106.41 105.67 104.65 103.26 101.37 98.877  
32 95.672 91.750 87.305 82.874 79.413 77.941

33 264 PERIOD QAT= 150626. QST= 8095242. QOT= 239995. HCI=1.7653 HCO=1.2370  
34 TAF= 130.84  
35 T1, T2, T3, -----T(M+1)  
36 107.95 107.22 106.19 104.80 102.90 100.38  
37 97.135 93.130 88.540 83.884 80.137 78.378

38 288 PERIOD QAT= 154188. QST= 8789580. QOT= 313526. HCI=1.7734 HCO=1.2520  
39 TAF= 132.22  
40 T1, T2, T3, -----T(M+1)  
41 109.43 108.69 107.67 106.27 104.36 101.83  
42 98.543 94.467 89.750 84.893 80.874 78.820

43 312 PERIOD QAT= 157605. QST= 9472593. QOT= 398527. HCI=1.7812 HCO=1.2660  
44 TAF= 133.54  
45 T1, T2, T3, -----T(M+1)  
46 110.85 110.11 109.09 107.68 105.77 103.22  
47 99.903 95.765 90.936 85.897 81.619 79.264

48 326 PERIOD QAT= 160896. QST= 10144026. QOT= 45171. HCI=1.7987 HCO=1.2720





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-2

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO.

43

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-88	J. Jones	6/13/88	1				

1

TAF= 134.82

T1,T2,T3, ----- T(M+1)

112.22 111.48 110.46 109.05 107.13 104.57  
101.22 97.028 92.099 86.893 82.367 79.708

12 HR 360 PERIOD QAT= 164075. QST=10803951. QOT= 603557. HCI=1.7960 HCO=1.2910

TAF= 136.06

T1,T2,T3, ----- T(M+1)

113.55 112.81 111.78 110.37 108.45 105.87  
102.50 98.259 93.240 87.880 83.113 80.148

432 PERIOD QAT= 173055. QST=12713487. QOT= 999325. HCI=1.8170 HCO=1.3229

TAF= 139.57

T1,T2,T3, ----- T(M+1)

117.31 116.57 115.54 114.12 112.19 109.58  
106.15 101.78 96.533 90.763 85.319 81.437

504 PERIOD QAT= 181352. QST=14519037. QOT= 1499768. HCI=1.8368 HCO=1.3495

TAF= 142.83

T1,T2,T3, ----- T(M+1)

120.81 120.07 119.03 117.61 115.66 113.03  
109.55 105.09 99.643 93.516 87.441 82.664

576 PERIOD QAT= 189088. QST=16223610. QOT= 2101749. HCI=1.8555 HCO=1.3723

TAF= 145.89

T1,T2,T3, ----- T(M+1)

124.08 123.34 122.31 120.88 118.92 116.27  
112.74 108.19 102.58 96.128 89.458 83.818

648 PERIOD QAT= 196333. QST=17831216. QOT= 2801153. HCI=1.8733 HCO=1.3921

TAF= 148.77

T1,T2,T3, ----- T(M+1)

127.16 126.42 125.38 123.95 121.98 119.31  
115.75 111.12 105.35 98.597 91.366 84.899

24 HR 720 PERIOD QAT= 203133. QST=19346347. QOT= 3593460. HCI=1.8903 HCO=1.4095

TAF= 151.48

T1,T2,T3, ----- T(M+1)

130.06 129.32 128.28 126.85 124.87 122.18  
118.58 113.88 107.97 100.93 93.163 85.911

27

28

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30

&gt;

31

32

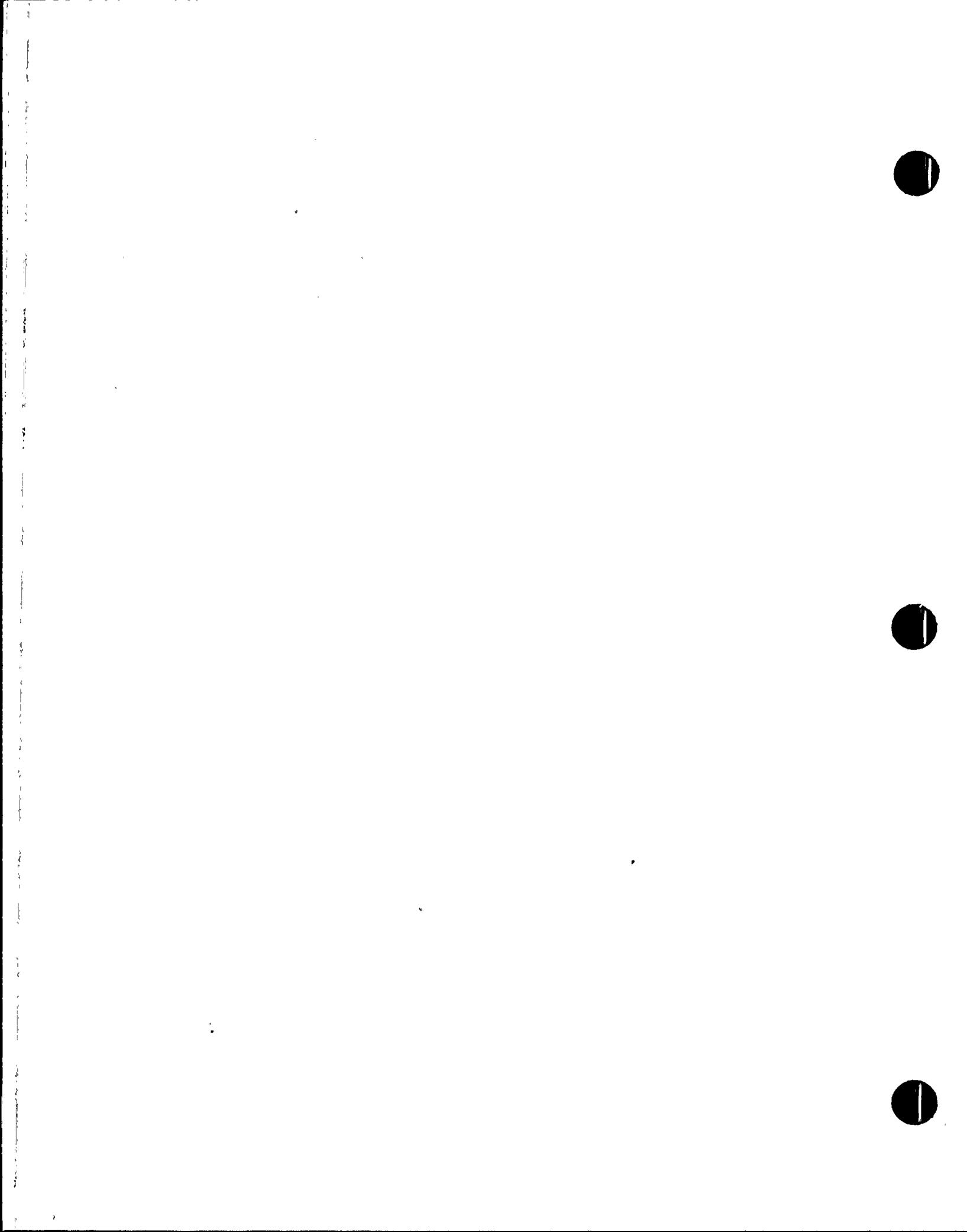
33

34

35

36

REV.





# CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-25

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 44

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE	REV
0	PSS	6-13-88	Dunn	6/3/88	1					

## APPENDIX C

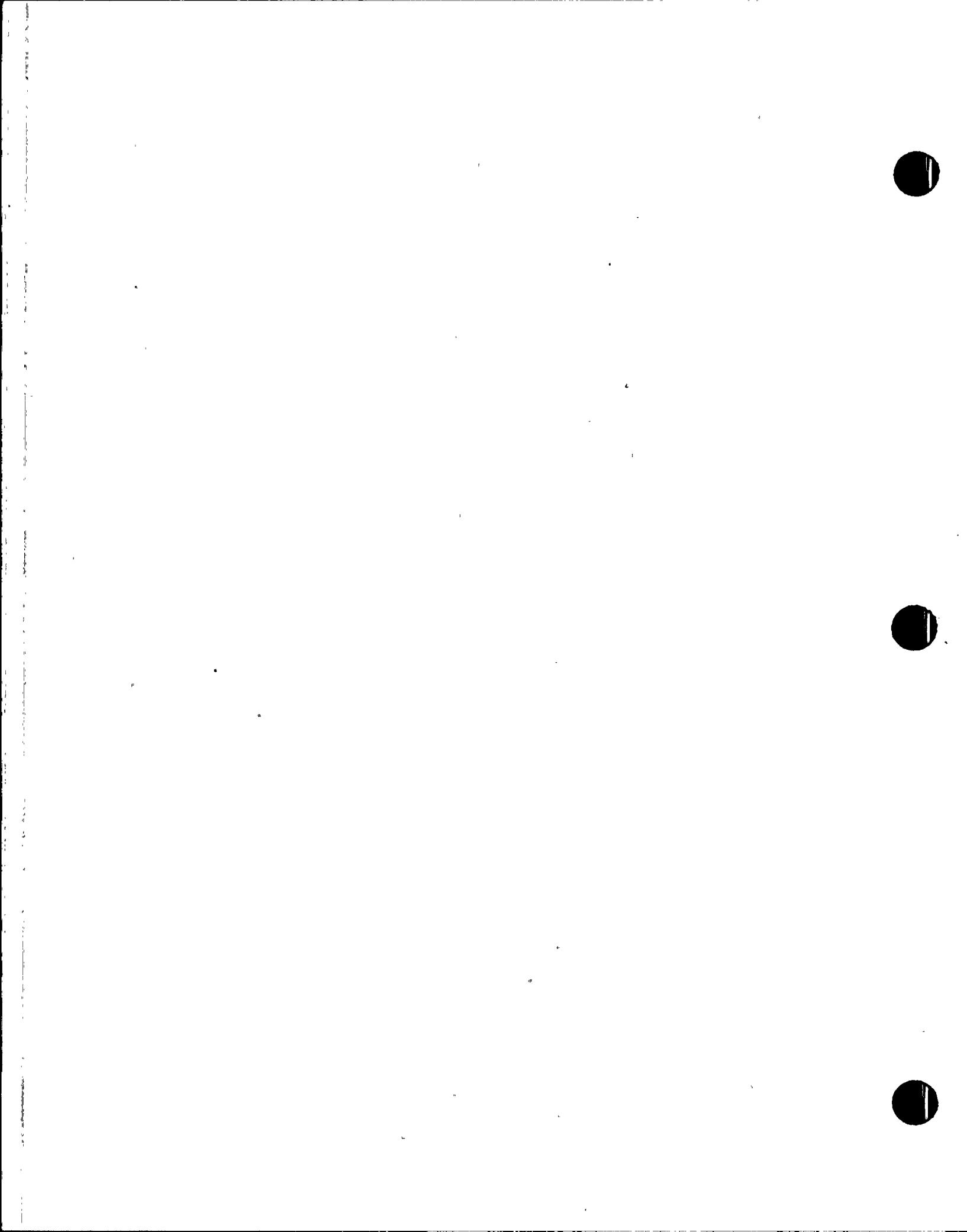
- INITIAL INTERNAL ROOM AMBIENT TEMPERATURE, DEG. F=?  
 >75.5  
 INITIAL ADJACENT ROOM AMBIENT TEMPERATURE, DEG. F=?  
 >75.5  
 EQUIPMENT HEAT GENERATED IN THE INTERNAL ROOM, BTU/HR=?  
 >690375  
 NET INTERNAL ROOM SURFACE AREA, SQ. FT.=?  
 >23754  
 NET INTERNAL ROOM VOLUME, CU.FT.=?  
 >159973  
 THICKNESS OF ROOM ENCLOSURE, FT.=?  
 >1  
 DENSITY OF ROOM ENCLOSURE MATERIAL, LBS/CU.FT.=?  
 >144  
 THERMAL CONDUCTIVITY OF ROOM ENCLOSURE MATERIAL, BTU/HR-FT-F=?  
 >.54  
 SPECIFIC HEAT OF ROOM ENCLOSURE MATERIAL, BTU/LB-F=?  
 >.2  
 ONE PERIOD OF TIME INCREMENT FOR CALCULATION, MIN.=?  
 >2  
 IMAGINARY THICKNESS OF FIRST LAYER OF ROOM ENCLOSURE, FT.=?  
 >.01  
 MULTIPLICATION FACTOR OF IMAGINARY THICKNESS OF OTHER LAYERS=?  
 >1.41

\*\*\*\*\*  
 \* COPYRIGHT 1976, 1979 BECHTEL POWER CORPORATION. ALL RIGHTS RESERVED. \*  
 \*\*\*\*\*

M=NUMBER OF IMAGINARY LAYER= 11

DX1, DX2, DX3, -----DX(M)					
1.00000-02	1.41000-02	1.98810-02	2.80322-02	3.95254-02	5.57308-02
7.85805-02	.11080	.15623	.22028	.26685	

21111	1 PERIOD QAT= 17078. QST= 5934. QOT= 0. HCl=1.3185 HCo=.0000
	TAF= 81.49
	T1, T2, T3, -----T(M+1)
	75.805 75.708 75.622 75.560 75.523 75.506
	75.501 75.500 75.500 75.500 75.500 75.500
	2 PERIOD QAT= 29400. QST= 16624. QOT= 0. HCl=1.4002 HCo=.0000
	TAF= 85.86
	T1, T2, T3, -----T(M+1)
	76.216 76.024 75.838 75.684 75.579 75.525
	75.505 75.501 75.500 75.500 75.500 75.500





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-25

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 45

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-88	Morrie	6/13/88	0				
					0				

REV.

3 PERIOD QAT= 38244. QST= 30792. QOT= 0. HCI=1.4468 HCO=.0000  
TAF= 89.02  
T1,T2,T3, -----T(M+1)

76.652 76.385 76.108 75.857 75.669 75.560  
75.514 75.502 75.500 75.500 75.500 75.500

6 PERIOD QAT= 52704. QST= 85368. QOT= 0. HCI=1.5099 HCO=.9468  
TAF= 94.23  
T1,T2,T3, -----T(M+1)

77.863 77.463 77.002 76.519 76.081 75.760  
75.583 75.517 75.502 75.500 75.500 75.500

12 PERIOD QAT= 62267. QST= 213877. QOT= 0. HCI=1.5417 HCO=.9520  
TAF= 97.71  
T1,T2,T3, -----T(M+1)

79.635 79.158 78.562 77.859 77.108 76.415  
75.899 75.620 75.522 75.502 75.500 75.500

18 PERIOD QAT= 66301. QST= 347915. QOT= 1. HCI=1.5512 HCO=.9589  
TAF= 99.19  
T1,T2,T3, -----T(M+1)

80.904 80.409 79.771 78.984 78.081 77.154  
76.355 75.822 75.577 75.510 75.501 75.500

24 PERIOD QAT= 69175. QST= 483108. QOT= 5. HCI=1.5573 HCO=.9669  
TAF= 100.24  
T1,T2,T3, -----T(M+1)

81.933 81.430 80.772 79.941 78.950 77.874  
76.860 76.093 75.674 75.530 75.502 75.501

1 HR 30 PERIOD QAT= 71574. QST= 618769. QOT= 16. HCI=1.5622 HCO=.9754  
TAF= 101.12  
T1,T2,T3, -----T(M+1)

82.821 82.314 81.643 80.784 79.736 78.554  
77.376 76.405 75.806 75.563 75.505 75.504

36 PERIOD QAT= 73687. QST= 754700. QOT= 43. HCI=1.5664 HCO=.9843  
TAF= 101.90  
T1,T2,T3, -----T(M+1)

83.615 83.105 82.426 81.546 80.457 79.196  
77.887 76.741 75.967 75.612 75.511 75.508

42 PERIOD QAT= 75600. QST= 890807. QOT= 96. HCI=1.5703 HCO=.9932  
TAF= 102.60  
T1,T2,T3, -----T(M+1)

84.340 83.827 83.142 82.247 81.125 79.802  
78.387 77.090 76.151 75.675 75.521 75.514

48 PERIOD QAT= 77360. QST= 1027027. QOT= 187. HCI=1.5738 HCO=1.0021  
TAF= 103.25  
T1,T2,T3, -----T(M+1)

85.011 84.496 83.806 82.899 81.752 80.378  
78.875 77.445 76.352 75.751 75.535 75.523

2 HR 60 PERIOD QAT= 80540. QST= 1299630. QOT= 548. HCI=1.5801 HCO=1.0194  
TAF= 104.42





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-25

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO.

46

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE	REV
0	PSS	6-13-68	Haner	6/13/68	1					
1					2					

T1, T2, T3, -----T(M+1)  
 86.230 85.713 85.015 84.091 82.904 81.453  
 79.809 78.160 76.791 75.939 75.578 75.552

72 PERIOD QAT= 83383. QST= 1572213. QOT= 1264. HCI=1.5858 HCO=1.0360  
 TAF= 105.47

T1, T2, T3, -----T(M+1)  
 87.328 86.809 86.105 85.167 83.952 82.443  
 80.692 78.867 77.260 76.165 75.642 75.594

84 PERIOD QAT= 85978. QST= 1844538. QOT= 2488. HCI=1.5910 HCO=1.0516  
 TAF= 106.44

T1, T2, T3, -----T(M+1)  
 88.334 87.813 87.105 86.157 84.920 83.366  
 81.530 79.560 77.746 76.421 75.729 75.650

96 PERIOD QAT= 88379. QST= 2116390. QOT= 4377. HCI=1.5958 HCO=1.0663  
 TAF= 107.33

T1, T2, T3, -----T(M+1)  
 89.268 88.746 88.035 87.079 85.824 84.233  
 82.328 80.236 78.240 76.699 75.837 75.721

108 PERIOD QAT= 90624. QST= 2387579. QOT= 7083. HCI=1.6004 HCO=1.0802  
 TAF= 108.16

T1, T2, T3, -----T(M+1)  
 90.144 89.621 88.907 87.944 86.674 85.054  
 83.089 80.894 78.736 76.995 75.967 75.804

41/R 120 PERIOD QAT= 92740. QST= 2657934. QOT= 10755. HCI=1.6047 HCO=1.0933  
 TAF= 108.95

T1, T2, T3, -----T(M+1)  
 90.971 90.448 89.731 88.763 87.480 85.834  
 83.819 81.534 79.232 77.304 76.116 75.900

132 PERIOD QAT= 94745. QST= 2927299. QOT= 15527. HCI=1.6087 HCO=1.1056  
 TAF= 109.70

T1, T2, T3, -----T(M+1)  
 91.757 91.233 90.514 89.541 88.248 86.580  
 84.521 82.156 79.725 77.626 76.282 76.006

144 PERIOD QAT= 96656. QST= 3195530. QOT= 21527. HCI=1.6126 HCO=1.1172  
 TAF= 110.41

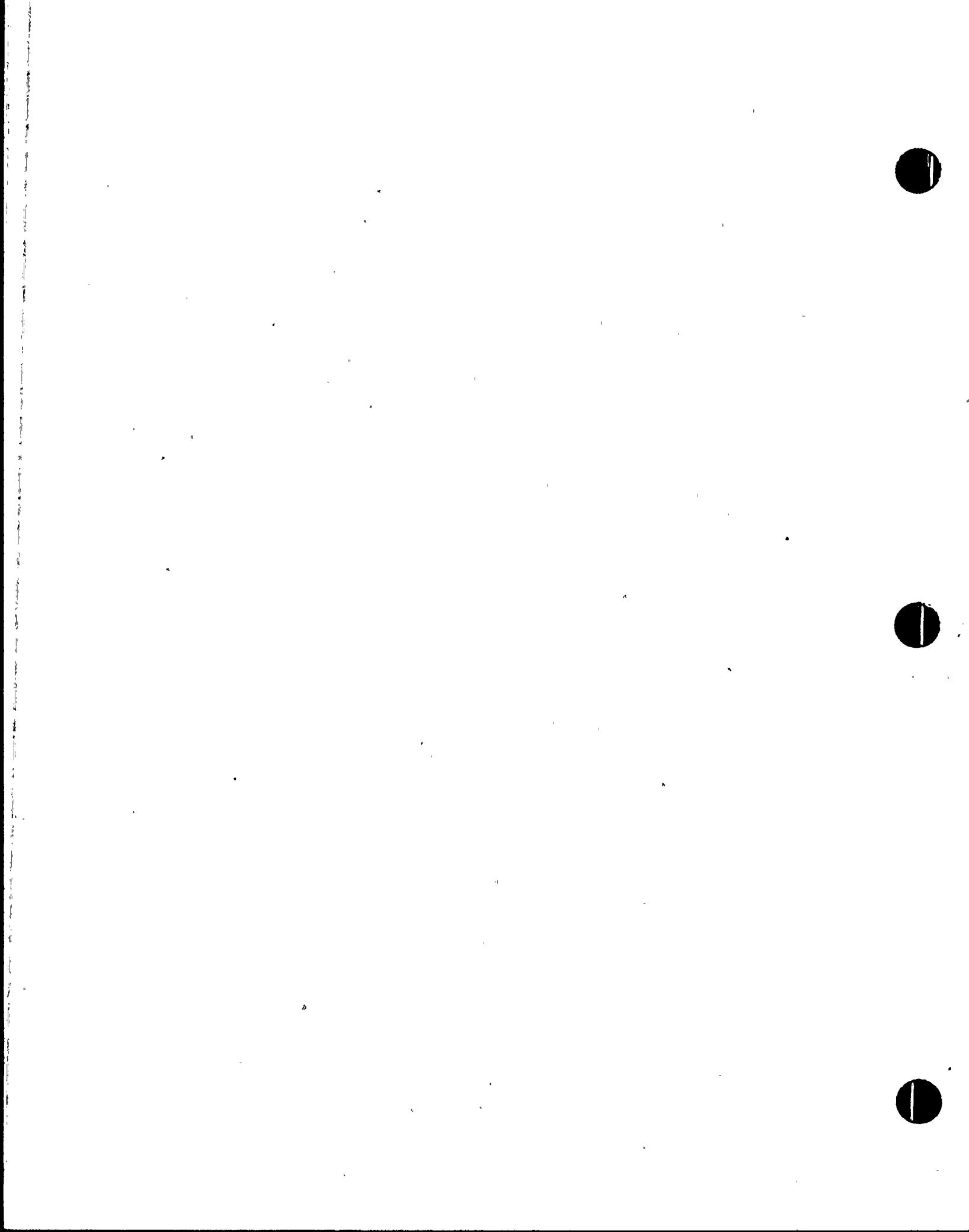
T1, T2, T3, -----T(M+1)  
 92.507 91.982 91.262 90.285 88.982 87.294  
 85.198 82.761 80.213 77.956 76.465 76.121

156 PERIOD QAT= 98483. QST= 3462504. QOT= 28869. HCI=1.6163 HCO=1.1282  
 TAF= 111.09

T1, T2, T3, -----T(M+1)  
 93.226 92.701 91.979 90.998 89.687 87.982  
 85.831 83.351 80.697 78.294 76.661 76.245

168 PERIOD QAT= 100237. QST= 3728103. QOT= 37656. HCI=1.6199 HCO=1.1386  
 TAF= 111.75

T1, T2, T3, -----T(M+1)  
 93.917 93.391 92.668 91.683 90.366 88.645  
 86.483 83.926 81.176 78.678 76.869 76.376





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-2E

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO.

47

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-88	J. Horner	6/13/88	1				
1					2				

1  
2  
3 6 HR 180 PERIOD QAT= 101926. QST= 3992231. QOT= 47982. HCI=1.6234 HCO=1.1484  
4 TAF= 112.38  
5 T1,T2,T3, -----T(M+1)  
6 94.583 94.057 93.332 92.345 91.020 89.286  
7 87.097 84.488 81.650 78.986 77.088 76.513  
8  
9  
10 192 PERIOD QAT= 103555. QST= 4254796. QOT= 59929. HCI=1.6268 HCO=1.1578  
11 TAF= 112.99  
12 T1,T2,T3, -----T(M+1)  
13 95.227 94.700 93.974 92.984 91.654 89.907  
14 87.693 85.037 82.119 79.339 77.317 76.656  
15  
16 204 PERIOD QAT= 105131. QST= 4515719. QOT= 73569. HCI=1.6300 HCO=1.1667  
17 TAF= 113.58  
18 T1,T2,T3, -----T(M+1)  
19 95.850 95.323 94.596 93.604 92.268 90.510  
20 88.273 85.574 82.582 79.695 77.553 76.802  
21  
22 216 PERIOD QAT= 106659. QST= 4774933. QOT= 88968. HCI=1.6332 HCO=1.1752  
23 TAF= 114.15  
24 T1,T2,T3, -----T(M+1)  
25 96.455 95.928 95.200 94.206 92.865 91.097  
26 88.839 86.101 83.041 80.053 77.796 76.953  
27  
28 228 PERIOD QAT= 108143. QST= 5032375. QOT= 106180. HCI=1.6363 HCO=1.1833  
29 TAF= 114.71  
30 T1,T2,T3, -----T(M+1)  
31 97.044 96.516 95.788 94.791 93.446 91.668  
32 89.392 86.617 83.495 80.412 78.045 77.106  
33  
34 8 HR 240 PERIOD QAT= 109587. QST= 5287995. QOT= 125256. HCI=1.6393 HCO=1.1911  
35 TAF= 115.26  
36 T1,T2,T3, -----T(M+1)  
37 97.617 97.089 96.360 95.361 94.012 92.226  
38 89.932 87.125 83.944 80.773 78.299 77.262  
39  
40 264 PERIOD QAT= 112368. QST= 5793589. QOT= 169160. HCI=1.6451 HCO=1.2057  
41 TAF= 116.30  
42 T1,T2,T3, -----T(M+1)  
43 98.723 98.194 97.464 96.462 95.105 93.304  
44 90.979 88.113 84.828 81.496 78.818 77.579  
45  
46 288 PERIOD QAT= 115023. QST= 6291427. QOT= 220945. HCI=1.6507 HCO=1.2191  
47 TAF= 117.30  
48 T1,T2,T3, -----T(M+1)  
49 99.781 99.252 98.520 97.515 96.153 94.338  
50 91.987 89.070 85.694 82.219 79.348 77.901  
51  
52 312 PERIOD QAT= 117570. QST= 6781304. QOT= 280800. HCI=1.6561 HCO=1.2314  
53 TAF= 118.27  
54 T1,T2,T3, -----T(M+1)  
55 100.80 100.27 99.535 98.528 97.161 95.335  
56 92.961 89.999 86.543 82.938 79.883 78.225  
57  
58





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-25

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 48

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
10	PSS	5-13-88	J. Horne	6/13/88	1				
11					1				

REV.  
A/INDI

TAF= 119.20

T1, T2, T3, -----T(M+1)

101.78 101.25 100.52 99.507 98.134 96.298  
93.904 90.904 87.376 83.652 80.421 78.548

12HR 360 PERIOD QAT= 122397. QST= 7736676. QOT= 425159. HCI=1.6664 HCO=1.2537

TAF= 120.10

T1, T2, T3, -----T(M+1)

102.73 102.20 101.46 100.45 99.077 97.233  
94.821 91.785 88.193 84.359 80.958 78.870

432 PERIOD QAT= 129104. QST= 9108150. QOT= 703796. HCI=1.6808 HCO=1.2819

TAF= 122.65

T1, T2, T3, -----T(M+1)

105.42 104.89 104.16 103.14 101.75 99.888  
97.432 94.310 90.553 86.428 82.546 79.814

504 PERIOD QAT= 135311. QST= 10406394. QOT= 1056153. HCI=1.6944 HCO=1.3056

TAF= 125.02

T1, T2, T3, -----T(M+1)

107.93 107.40 106.66 105.64 104.24 102.36  
99.870 96.676 92.783 88.405 84.078 80.714

576 PERIOD QAT= 141107. QST= 11633488. QOT= 1480075. HCI=1.7072 HCO=1.3257

TAF= 127.24

T1, T2, T3, -----T(M+1)

110.27 109.74 109.00 107.98 106.58 104.68  
102.16 98.905 94.891 90.283 85.537 81.564

648 PERIOD QAT= 146544. QST= 12792212. QOT= 1972721. HCI=1.7194 HCO=1.3433

TAF= 129.33

T1, T2, T3, -----T(M+1)

112.48 111.95 111.21 110.19 108.78 106.87  
104.32 101.01 96.882 92.060 86.918 82.363

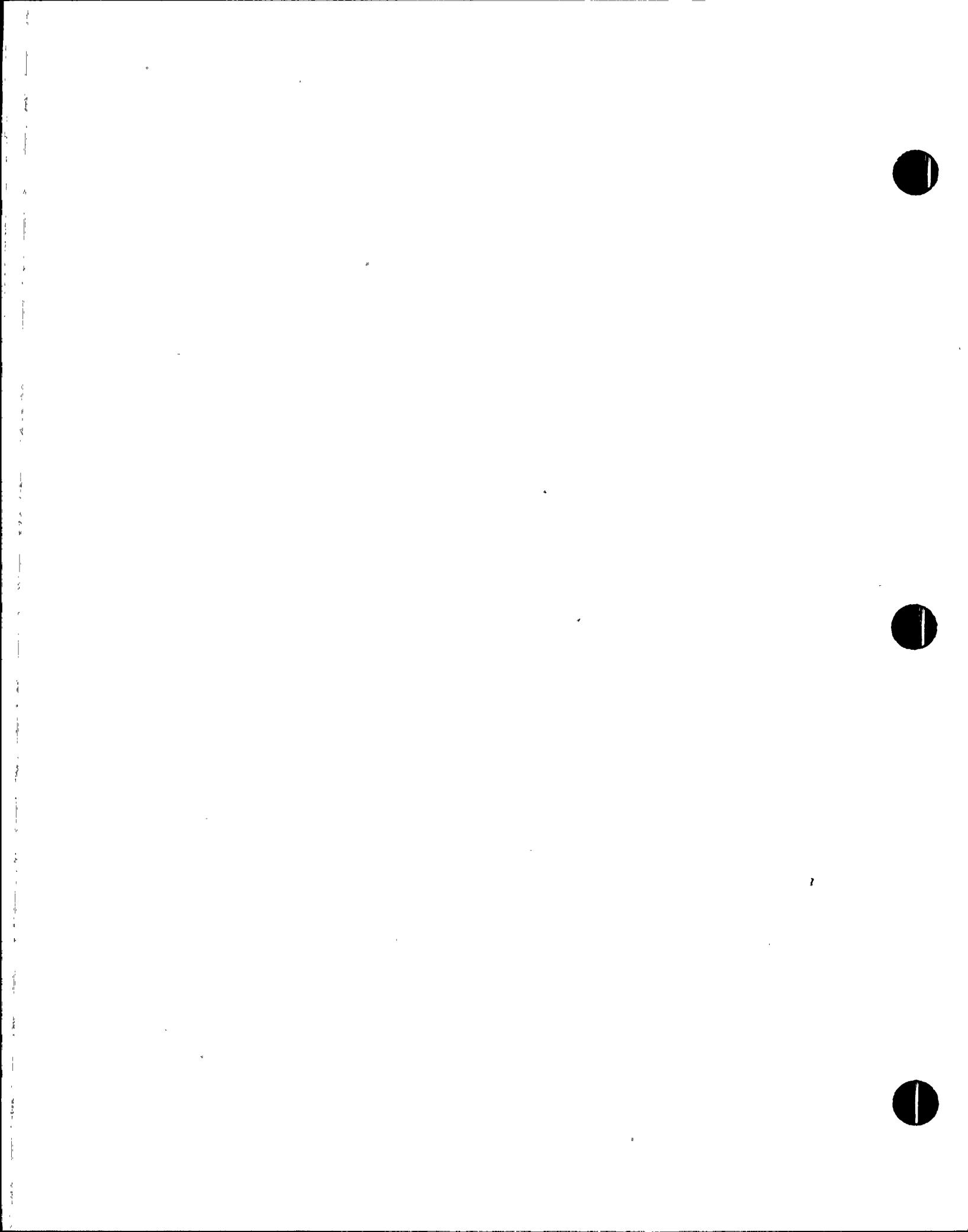
24HR 720 PERIOD QAT= 151656. QST= 13885662. QOT= 2530963. HCI=1.7310 HCO=1.3586

TAF= 131.31

T1, T2, T3, -----T(M+1)

114.57 114.03 113.29 112.26 110.85 108.93  
106.35 102.99 98.763 93.741 88.222 83.111

&gt;





## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-253

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO.

49

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
10	PSS	6-13-88	J. Jones	6/13/88	1				

## APPENDIX D

EA

Sheet 2 of 3

**ENGINEERING EVALUATIONS**  
**PALO VERDE NUCLEAR GENERATING STATION**  
**ENGINEERING ANALYSIS WORK SHEET**









## CALCULATION SHEET

PROJECT ANPP

JOB NO. 18601 - 200

CALC. NO. 13-MC-HJ-2

SUBJECT TRANSIENT TEMPERATURE STUDY FOR MAIN CONTROL ROOM

SHEET NO. 51

REV	ORIGINATOR	DATE	CHECKER	DATE	REV	ORIGINATOR	DATE	CHECKER	DATE
0	PSS	6-13-88	Horus	6/13/88	0				

EA  
Sheet 2 of 2ENGINEERING EVALUATIONS  
PALO VERDE NUCLEAR GENERATING STATION  
ENGINEERING ANALYSIS WORK SHEET

Title: METEOROLOGICAL DATA TAKEN FROM MET TOWER CHARTS  
FOR 5/20/88 THRU 5/29/88 AND 6/9/88  
Performed by: R.A.ZARBO Date: 6/10/88

References:	Review Method by:	Alternate Calc.
		<input type="checkbox"/>
		<input type="checkbox"/>
		<input type="checkbox"/>
Reviewed by: DMR		
DATE HIGH TEMP (°F)		
5/20/88	93.9	
5/21/88	96.6	
5/22/88	100.1	
5/23/88	99.9	
5/24/88	98.4	
5/25/88	99.1	
5/26/88	101.3	
5/27/88	97.0	
5/28/88	94.2	
5/29/88	78.9	
6/9/88	99.8	

