WESTINGHOUSE VALVE PRESSURE LOCKING SPECIAL TEST PROCEDURE

Revision 0 September 12, 1995

ComEd Company

Corporate MOV Program Support

Prepared by:

9605310074 9605 PDR ADDCK 0500

0237 PDR John F. Kelly MOV Program Support

Approved by:2 Ivo Garza

NES MOV Program Engineer

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A. <u>PURPOSE</u>

The purpose of this special test is to validate the proposed model and input assumptions for quantifying capability margin for valves susceptible to pressure locking. Specifically, testing will verify:

- the model for estimating MOV pressure lock pullout forces
- bonnet ability to retain pressure when upstream pressure source is removed
- bonnet pressure response to temperature changes.

The MOV for this special test is a Westinghouse valve. This procedure provides the test requirements, procedures, and equipment to be used.

This special test procedure shall not be used on equipment installed in an operating Nuclear Station.

B. <u>REFERENCES</u>

- 1. Generic Letter 95-07, Pressure Locking and Thermal Binding
- 2. ComEd Quality Assurance Program Sections 9 and 11

C. TEST EQUIPMENT AND INSTRUMENTATION

- All instrumentation, measuring, and test equipment used in the performance of this test program should be calibrated in accordance with ComEd's Quality Assurance Program
- 2. Measurement Equipment is listed in Table 1
- 3. Thrust, torque, motor power, and motor current shall be monitored.
- 4. Upstream, downstream, and bonnet pressure and temperature should be recorded as specified herein.
- 5. Westinghouse valve
- 6. Teledyne Quick Stem Sensor
- 7. Hydro-pump capable of generating 2500 psi
- 8. Miscellaneous valves and fittings

D. PRECAUTIONS

- 1. Standard safe work practices shall be followed when working around high pressure and electrical test equipment.
- 2. Do not exceed 2500 psig pressure.

E. <u>REQUIREMENTS AND PROCEDURES</u>

Table 2 specifies the testing to be performed and the test sequence. This test sequence may be modified during the special test. New or revised test sequences should be added to Table 2.

- 1. Pre-Test Preparation
 - a. Record valve and actuator nameplate data into the test Data Sheet (Appendix A-7).
 - b. The required measurements and associated instruments to be installed are listed in Table 1.
 - c. The data acquisition system will consist of the VOTES computer, breakout box, motor power monitor, and associated cables.
 - d. Pressures and temperatures will be recorded manually.
 - e. Prior to any testing or stroking of the valve, actuator switches shall be set as follows:

1) The open limit switch shall be set to prevent back-seating of the valve. 93%

A Nel 109-13-95 Signature Date

2) The open torque switch should be bypassed a minimum of 25% of the open travel distance.

109-13-95

- f. If necessary, calibration of the VOTES Force Sensor and Teledyne Quick Stem Sensor shall be documented on Attachment A-1
- 2. Static Break-in Test

Verify that the valve has been stroked a minimum of 15 strokes open and 15 strokes closed. If not, cycle valve until the specified strokes are achieved. $T_{A} = 13.57 \text{ Kw}$

2 109-13-95

3. LLRT Test

An LLRT Leakage Rate Test shall be performed at specified torque switch settings in both directions to verify seat leakage requirements in accordance with approved station procedures. Document results in Appendix A-2

4. Differential Pressure Test

- a. With the valve open fill the specimen with water .
- b. With the valve unpressurized, stroke test specimen open and then closed at the lower torque switch setting and record test data.
- c. Pressurize upstream disk side per Table 2.
- d. Vent downstream disk side to atmosphere.
- e. Open the valve, record diagnostic test data, and record upstream pressure.
- f. With the valve unpressurized, stroke test specimen closed and record test data.
- g. Perform valve factor calculation as described in Attachment A-3 and record results.
- 5. Bonnet Pressure Response
 - a. With the valve open fill the specimen with water such that air pockets are vented and bonnet is filled with water.
 - b. With the valve unpressurized and setup per Table 2, stroke test specimen open and then closed and record test data.
 - c. Pressurize valve bonnet to the pressure indicated in Table 2 for this test.
 - d. Vent upstream and downstream disk side to atmosphere and record bonnet pressure as a function of time.
- 6. Pressure Lock Test
 - a. With the valve open fill the specimen with water such that air pockets are vented and bonnet is filled with water.
 - b. With the valve unpressurized and setup per Table 2, stroke test specimen open and then closed and record test data.
 - c. Pressurize bonnet to the pressure indicated in Table 2 for this test.
 - d. Vent downstream and upstream disk side to atmosphere.
 - e. Record bonnet pressure and open/close the valve while recording diagnostic test data.

7. Bonnet Pressure Response to Temperature Changes

- a. With the valve open fill the specimen with water such that air pockets are vented and bonnet is filled with water.
- b. With the valve unpressurized and setup per Table 2, stroke test specimen open and then closed and record test data.
- c. Heat bonnet to maximum achievable temperature.
- d. Monitor and record fluid temperature and bonnet pressure in Appendix A-6 until stable.

F. RESULTS/ACCEPTANCE CRITERIA

The results of this test will be used as technical input for evaluations and calculations to resolve/assess the pressure locking issue. This test has no acceptance criteria. Verify that calibrated equipment is within calibration dates at the completion of testing.

mpm ral out of date. will be relibertil SULSEQUENT to test.

G. <u>RECORDS</u>

The following information shall be retained as Quality Records:

- Diagnostic Test Data, VOTES and MPM (electronic format)
- Training Qualification Records for personnel operating test equipment
- Test results Data Sheets
- Test Procedure

H. PERSONNEL QUALIFICATIONS

Operation of diagnostic test equipment shall be performed by qualified individuals. This includes:

- Liberty Technologies Basic, Advanced, and Continuing VOTES System Training
- Liberty Technologies Advanced MPM Training

Personnel performing diagnostic testing shall sign and date the applicable Data Sheet(s)

I. <u>SCHEDULE</u>

A specific completion date is not required for this test.

TABLE 1 CALIBRATION LOG MEASUREMENT EQUIPMENT AND TOLERANCES

Measurement Parameter	Device Name	QA/Serial #	Calibration Date/Due Date
Pressure Gage Upstream Disk Side	GAGE	66295	9-7-95/Rost Test
Pressure Gage Downstream Side		Not Use	2
Pressure Gage Bonnet	GAGE	60900	9-7-95/PostTest
Temperature Gage Bonnet		NOT USED -	· · ·
Stem Torque	Teledyne Quick Stem Sensor	-	_
Stem Torque	Liberty , VTC		
Stem Thrust	Teledyne Quick Stem Sensor	_	
Stem Thrust (Verification)	Liberty, C-Clamp	278973BR	07-95/01-96
Motor Power Monitor System	Liberty, MPM	A1019/61020	7-23-94/1-23-95
VOTES System Compositor	Liberty		
VOTES Breakout Box	Liberty		07-95/01-96
I. BI	MPM Compter	A1014/ C1020	7-23-94/1-23-95

LLRI

Pressure

Flow Metri MPM OPERATOR : VOTES OPERATOR: Chris Bedford

LLRÍ XE lister ote John Kelly

LLAT

104945BR 12-94/12-95

033200BR 8195/2-96

Frepured

Reviewed

09-13

Date

Date

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TABLE 2
TESTING SEQUENCE AND NUMBERING

Procedure Section	
F.3	LLRT1
F.4	Differential pressure test to quantify friction factor at 500 psi
F.4	Differential pressure test to quantify friction factor at 1000 psi
F.4	Differential pressure test to quantify friction factor at 1500 psi
F.5	Bonnet Pressure Response at 1000 psi and lowest torque switch setting
F.5	Bonnet Pressure Response at 2000 psi and lowest torque switch setting
F.5	Bonnet Pressure Response at 1000 psi and highest torque switch setting
F.5	Bonnet Pressure Response at 2000 psi and highest torque switch setting
F.6	Pressure Lock Un-wedging at 500 psi and lower torque switch setting
F.6	Pressure Lock Un-wedging at 1000 psi and lower torque switch setting
F.6	Pressure Lock Un-wedging at 1500 psi and lower torque switch setting
F.6	Pressure Lock Un-wedging at 2000 psi and lower torque switch setting
	5
F.6	Pressure Lock Un-wedging at 500 psi and higher torque switch setting
F.6	Pressure Lock Un-wedging at 1000 psi and higher torque switch setting
F:6	Pressure Lock Un-wedging at 1500 psi and higher torque switch setting
F.6	Pressure Lock Un-wedging at 2000 psi and higher torque switch setting
F.7	Bonnet pressure response to temperature at higher torque switch setting.

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QSS ₩FS CALIBRATION FIELD DATA SHEET 09/12/95 Page 9 of 15

Appendix A1

VALVE TAG NUMBER: TEST VALUE		VOTES SYSTEM SERIAL NO .: A 106 8				
VOTES SYSTEM QA NO .: 778951 BR		CAL DUE DATE:	1/96			
CALIBRATOR LOCATION: THREADED	N-THREADED SLOTTED	TRANSITION				
DESCRIPTION: VOTES SYSTEM W CALIBRATOR USED TO CALIB		023 3021D	SECTION EFFECTION DIA 1.229			
NEW EFFECTIVE STEM DIA. 1.229	CB3-100 LENGTH: Jo ft		AMP PROBE SETTING: 2V 20A			
ANTI-ROTATION DEVICE: (yes) no						

CALIBRATION TABLE

RUN #	Test Number/Date	votes sens No. ८९८८	CAL DEV. NO.	CLAMP PRE- TENSION READING	ŤSS	MAX _ THRUST	RSQ	CFA	BFSL SENS	BFSL % CHG	STEM TEMP (F)	GAIN
	9/13/95	3352	20023	2990	1.0	13532	1,000	1.000	-2.43750		704	515
2	9115/45	11	10023	2995	2.0	19,819	1.000	1.000	-2.44725	0.41	70°F	515
3		· · ·		· · · · · · · · · · · · · · · · · · ·		 						
						•	·					
	· ·											

Prepared: Date: 9/13/95 Reviewed: RCBd Date: 9/13/95

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LLRT RESULTS DATA SHEET

Appendix A2

VOTES Test #/Date	TSS	C14, lbf	Cl6, lbf	Pullout, Ibf	Leakage, scfm	Comments, Note upstream or downstream test.
10/9-13	1.5	14,877	15,447	-	0 *	UPSTREAM SIDE TEST
	1.5				0 *	DOWNSTREAM SIDE TEST
11/9-13	1.0	13,183	13,820		0*	oppisite flow direction - Downsrranm
	0.1				0 ×	UPSTREAM SIDE FEST
	-				-	

Date: 9-13-95 Reviewed: 26. Blf Date: 9/13/95 Prepared:

& VALUES ARE ACCURATE TO WITHIN G.4- SCFH





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PRESSURE LOCKING SPECIAL TEST PROCEDURE Revision 0 DIFFERENTIAL PRESSURE TEST RESULTS DATA SHEET

<14/C16/57 VOTES Pullout Downstream 010 Upstream Open Open **Disk Side** Run Valve Test #/ Thrust. Thrust, **Disk Side** Thrust, Factor¹ Date/ lbf Pressure, Pressure, lbf[∗] Thrust, Comments lbf TSS lbf psi psi 14929 6 /9113 \bigcirc 150506, 3167135 500/460 0.33 2430 11015 1517 148101 2904 500/490 7 1.9113/ 1508 2359 0.314 \mathcal{O} 1465 +019 1040 81 9-13-95, 15347 3378 4456/980 2640 0) 1482 0.251 1.5:47 .258 15,012 1690/1590 9 3501 1541 \mathcal{O} 2675 15502 0,208 1980/1880 13079/ 13 \mathcal{O} 2727 1581 13751 3500 0.194 ٠.

¹Valve Factor = $\frac{O10 - \text{Run Load} + \left[\text{Upstream Pressure} \times \frac{\pi}{4} (1.25)^2 \right]}{\text{Upstream Pressure} \times \frac{\pi}{4} (3.445)^2}$ Prepared: $\frac{1}{4} \frac{1}{4} \frac{1}{4}$



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DIFFERENTIAL PRESSURE TEST RESULTS DATA SHEET

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mpm	VOTES Test #/ Date/ TSS	C16 Thrust, Ibf	Pullout Thrust, Ibf	Upstream Disk Side Pressure, psi	Downstream Disk Side Pressure, psi	O10 Thrust, Ibf	Open Run Thrust, Ibf	Open Valve Factor ¹	Comments		
	18	STATIC	TEST	510/505	0	2734					
	19	19269/ 1917	2751	310/505	0 2292	1713 2754	1582 w	-460.78			
	20	19252/1914	2570	1030/1020	0 2342		1615 9/0		<i>م</i> ک		
	21	19900	3091	1512/1496	0	2428 1679 une 1679 4.1595	1604	.19		,	
10:22	22	19351/19949	3125	1964/1944	0	2377		1.17			
10:42	23	19433 200AT	3108	1900/1880	O	2529	1694	.18	×		
10:54	24	195(3/ 20112	1	1596/1576	0	2359	1694	. 18	Run 10 1000 Roidel FTestmit	i)t>-	
11:00	25	19131/ /19946	2410	1078/1068	0	2070	1670	Note:	Opstrum pressures include	4	
	¹ Valve F	actor =	····	<u> </u>	m Pressure $\times \frac{\pi}{4}$ (1)	_ام			Initial Pressure white riching		
	Upstream Pressure × $\frac{\pi}{4}(3.445)^2$ Prepared: Date: 9-13-95 Reviewed: PC Ball Date: 9/15/95										
	Prepare	d: <u>{} } } }</u>	Da	te: 9-13-95	Reviewed: <u>P</u>	Ball	/ Date:	9/15/95			
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PRESSURE LOCKING SPECIAL TEST PROCEDURE Revision 0

DIFFERENTIAL PRESSURE TEST RESULTS DATA SHEET

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VOTES Test #/ Date/ TSS	C16 Thrust, Ibf	Pullout Thrust, Ibf	Upstream Disk Side Pressure, psi	Downstream Disk Side Pressure, psi	O10 Thrust, Ibf	Open Run Thrust, Ibf	Open Valve Factor ¹	Comments
46/01-15 2.0	14:477	2699	1532/1515	LEB AT 1315	20\$9	1742	0.15	
47/9.15	19410	3192	1564	0	2240	1754	0.16	Reverse Flow DP test
							· .	· · · · ·
					<u> </u>			

 $\frac{\pi}{1.25}^{2}$ O10 - Run Load + Upstream Pressure × ¹Valve Factor = Upstream Pressure × (3.445) Date: 9-13-95 Reviewed: 2CPrepared Date:





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Revision 0 PRESSURE LOCK TEST RESULTS DATA SHEET

Test Description	VOTES Test #/ Date	MPM Title/ Date/ TSS	Ci4/C16 Thrust, Ibf	09 Thrust, Ibf	Bonnet Pressure, psi	Pullout Motor Power, kW	Pullout Torque, Ibf	Comments
Static	15		19333/ 19835	1885	0	-	20.0	
Pressure Lock	16	4:23 1500 9-12 1500 PresLick TSS = 2.0	19946	5991	1650	1.8K	= 67.7	Inrus Bur= 13.61KW
Static	רו		14,147	1902	0		20.3	
				· .				
			· .	· .				
		· ·						
	_			<u> </u>				
	·						-	

Date: 9-13-95 Reviewed: R.C. Beak Date: 4/15/105 Prepared: 29/29

PRESSURE LOCKING SPECIAL TEST PROCEDURE Revision 0 PRESSURE LOCK TEST RESULTS DATA SHEET

L14/C16 09 Pullout Pullout Test Description VOTES MPM Title/ Bonnet Torque, Test #/ Date/ TSS Thrust. Thrust. Pressure. Motor Comments lbf Power, kW lbf Date lbf psi Sighic Prior to DP 20.7 C14 19503 STATE TOSIDAX 20101 1952 1.1441 Kw 18 0 11:07 19433 STATIC 26 1900 0 21.3 Studic OLP 17003 TSSMAX 1.17 11:13 1948/ Static 27 1866 Stadic OLP 0 1,20124 1.165 21.5 TSSMAX 11:19 19481/ Sitzie NOT 28 1441 16.6 Static 500 LP 500 TSSMAX 120179 TAKEN 11:21 19432, NOT Static 1458 17.3 29 Static SOOLP 20080 500 TSSMAX TAKEN 11:26 19415/ 496 Classed value with TSSNAT PL at 500 100 bonnet 30 SALA/ 3005 1.165 KW 36.1 20096 PL 500 496500 PSI LP TSSMAX 11:36 14,414 11 PLi at 500/wbonct 31 514 1.194Kw 35.8 2988 19874 PL500 TSS MAX 11:45 11,351/ 498 Static 1000 LP 896 32 STATIC 1000LP 'SIK 10.4 19962 Closed vilve with TSS MAX 11:50 19,381, 978 47.5 978 051 2009/10/10 0,967 PL 44 1000 /20024 3907 33 1P PL 1000 TSS MAY 11:56 14,281 1P 1008 Static 1000LP 930 1008 10.7 0.789KW 34 19,962 STANIC 1000LP TSS -AX 12:01 19.380 988 3890 35 46.8 PL at 1000 1.367KW PL 1000 14.994 Src Tish 45 for TSMAX 12:06 18,001, Static 1500 LP 36 522 .692 2.2 1580 19,279 らちょもに 2-10 Date: 2-13-95 Reviewed: (LC Buch Date: 1/15/95 Prepared



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PRESSURE LOCK TEST RESULTS DATA SHEET

Test Description	VOTES Test #/ Date	MPM Title/ Date/ TSS	ر،ץ/C16 Thrust, Ibf	09 Thrust, Ibf	Bonnet Pressure, psi	Pullout Motor Power, kW	Pullout Torque, Ibf	Comments
PL 1500 psi	37	TSSMAX	19379	4825	1525	1.541	57.6	
STATIC 1500 LP	38	STATIC 12:22 TSSMAX	19,629	505	1590	0.75	20	Sec Tast 45 for Zero
PL 1500 psi	34	PL 1500 12:29 TSSMAX	14,411/20054	4722	1536	1.52	55.7 405	
STATIC 2000	40	Styfic TSEMAX	19,047	18381 296		0,679	NOT MKCN	At Closing _ 1862 OPEN <u>closed - 1980 RUB</u>
STATE 2000 Pt 2000 Million	41	STATIC. TSSMAX 2	19,677	-346	085221980 CLOSE=1980	0.735	0,0	00
PL 2000	42	PLZOO TSSMAX	29323	5589	1902	1.61 KW	65.3	
STATIC 2000	43	STATIC TOSMAX	19.976	-536	1920	NOT TAKEN	-0.2	
FL 2000	44	PL2000	19,544	5726	1950	1.575Kw	66.5	
1500 518 2006	45	STATIC TSSMAX	14,343	. 17	Орен = 150 0 close = 1580	· · · · ·	0.3	
FL-1500	RCB of spar	-						
			· · ·		а,			
Prepared: John	∑ Date:	9 <u>-13-95</u> Revie	ewed: <u>R</u>	Belf	_ Date: <u>1/1</u>	145		

PRESSURE LOCKING SPECIAL TEST PROCEDURE 9/12/95 **Revision** 0 Page 14 of 15 1 4 15 45 Bornet 1/4/1/5/43 PRESSURE RESPONSE FO TEMPERATURE DATA SHEET Appendix A6 TSS: 1.0 VOTES Test #/Date: 11 /9-13 <14/C16 Thrust: 13,18/13,804 09 Thrust: الم/A Bonnet Temperature, °F Bonnet Pressure, Time psig Packin Load ${\mathcal O}$ 1000 900 800 26 58 700 600 1:45 500 2:07 400 4:51 Air bubles from packing 300 Lak off line 8:40 2000 1400 - \mathcal{O} 13 1900 25 1800 38 1700 51 1600 1500 1:05 1:19 1400 1:35 1300 Prepared: TRP Date: 9-73-98 Reviewed: 1/(Date: 9/15/95

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9/12/95

BONNET LUBANONE PRESSURE RESPONSE TO TEMPÉRATURE DATA SHEET	Appendix A6
VOTES Test #/Date: 14/9-13 TSS 1.0	·

09 Thrust: ________

CI4/C16 Thrust: 13,178/13,816

Time	Bonnet Pressure, psig	Bonnet Temperature, °F
1:52	1200	
2:11	1100	
2:34	1060	
3:01	900	
3:32	SoO	
4:14	ס <i>ס</i> ר	
Closed 10pm	al votes te	st # 14
0	2000	
19	1900	
39	1800	
59	1700	
1:19	1606	
1:40	1500	
2:03	1400	
2:26	1300	1
Prepared: JAlel D	ate: <u>9-13-95</u> Review	red: <u>R. Burk</u> Date: <u>9/15</u> /9

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	BONNET CUS dis 4	
PRESSURE RESPONSE	TO TEMPERATURE DATA SHEET	Appendix A6

VOTES Test #/Date: 15/9-13

TSS: <u>2.0</u>

O9 Thrust: <u>م/م</u>

C14 C16 Thrust: 19383/ 19869

Time	Bonnet Pressure, psig	Bonnet-Temperature, °F
2:57	1200	
3:20	1100	
3:53	10 00	
4:33	900	
4:36	890	
4:40	880	
4:45	058	
4:49	860	
4:53	820	
5:16	800	
Static Raise	TSS 2:00 ->	T 23 1 # 15
0	1980	
28	1930	<u></u>
1:40	1900	
2:10	/ 880	ſ'n
Prepared: At Leff Da	ate: <u>9-13-95</u> Review	red: <u>R.(. Brit</u> Date: <u>9/13</u> 2

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PRESSURE RESPONSE	BONNET RBdife TO TEMPERATURE	
VOTES Test #/Date:	5/9-1]	TSS:
09 Thrust:/A	Ci4	/C16 Thrust: <u>19383 / 1986</u> 9
Time	Bonnet Pressure,	Bonnet Temperature, °F
3:13	1840	
4:20	. 1800	
4:55	1780	
Close val	r equalite pre	27022
0	1730	
52	1720	
(:40	0171	
		< <u>;</u>
Prepared: <u>AA Pol</u> Da	ate: <u>4345</u> Review	ved: <u>2C.Bzef</u> Date: <u>9/13</u> /45

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PRESSURE RESPONSE TO TEMPERATURE DATA SHEET Appendix A6

VOTES Test #/Date: _/8,26,27, 49

TSS: <u>20</u>

09 Thrust: 1952, 1900, 1.866, 2239

C16 Thrust: 20/01, 20031, 20/29, 20648

Time	Bonnet Pressure, psig	Bonnet Temperature, °F		
N/A	102	75.8		
	106.5	91		
	93.7	105		
	105.7	120		
· · · · · · · · · · · · · · · · · · ·	124.5	185		
	128.2	195		
	131.2	206		
	1350	216.2		
	139.5	726		
	153.7	230		
	161.3	236		
	173.3	215		
	182.2	250		
	192.7	257		
	201.7	263		

Prepared: <u>RBrok JFK</u> Date: <u>9/15/15</u> Reviewed <u>D.C. Bed</u> Date: <u>9/15/15</u>

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VALVE DATA SHEET

Appendix A7

Va	Valve			
Туре	Gate			
Vendor/ID	Westinghuse/4-5M88F1 4" 2500psi/650°F	ND Nameplate		
Size	4" 2500 psi/650°F	Nanoplate		
Model No./Serial No.		0W750007 Namplate		
Mean Seat Diameter				
Stem Diameter				
Stem Threads per Inch				
Stem Thread Starts	2	, .		
Stem Material		· · · · · · · · · · · · · · · · · · ·		
YRBJIL	1978			
ASMB	CLASSI			
Actu	ator			
Туре				
Vendor	Lintorque	Name Plate / Octasha		
Size	53-00	<u> 11</u>		
Model No.				
Serial No	265405			
OAR	38.6	. [1		
Spring Pack No.	047	11		
Shop Order No	047 3B3749A	<u> </u>		
 Mot	or			
Туре				
Vendor	Relignce	Nameplate		
Model No. ID No	716244-VC	()		
RPM	001	11		
Voltage (AC/DC)	AC	()		
Start Torque	15 ft-15	11		
Run Torque	3 F+.15	11		
Locked Rotor Amps				
Full Load Amps				
Frame Size	M-56	Name plate		
RunAnps	2.8	11		
Insulation Class	RH	" 10		

Prepared: A Kith Date: 91395 Reviewed: R.C. But Date: 9/13/95 Weak Link 24,000 135, Teleron Steve Zong open and close

Actuato- 100% = 14,00015, 200% = 28,000155

Westinghouse Valve

Pressure Locking Thermal Binding Test Notes

09/13/95 Test Setup

The Westinghouse valve was received from the stand fabricator and was setup such that the valve could be rotated about the pipe centerline from vertical to 90 degrees from vertical. The valve leakoff line was machined and fitted with a cap which could be removed. The instrument maintenance department calibrated and installed the test equipment. Two holes were drilled and tapped into the bonnet to accept a thermowell/temperature meter and a pressure transducer/indicator. Due to the small size of the valve and the bonnet area and the length of the thermowell it could not be used for testing.

A high pressure air/water accumulator was used to pump high pressure water into either the upstream or downstream side of the valve. The accumulator would supply a constant water pressure during unseating of the valve.

Data Acquisition

The VOTES and MPM systems were used as data acquisition devices for the test. The VOTES system was used to. monitor stem thrust, switch actuation, and motor current. The MPM system was used to monitor motor voltage and current parameters. The Westinghouse valve has a solid unthreaded section of stem just above the antirotation device. In this area a Teledyne QSS was mounted and connected to the VOTES system. This QSS was then calibrated using a Liberty C-Clamp located just below the QSS 1" below the stem undercut. In this area the liberty effective diameter is 1.229 inches. A calibration was performed at a high valve torque switch setting of 2.0 and a low torque switch setting of 1.0. These two calibrations were within 0.41 percent of each other.

Local leak rate testing

A Local Leak Rate Test (LLRT) was performed in accordance with procedural step E.3 after initial differential pressure testing. This LLRT testing was performed in accordance with plant procedures with a test pressure of approximately 45.6 psig. The valve was tested on the upstream and downstream side at both a TSS of 1.0 and 2.0. Results of this testing indicated zero leakage in both directions. The test equipment has an accuracy 0.4 scfh.

09/13/95 Differential pressure testing

Differential pressure tests were performed on the upstream side of the valve at a TSS of 1.0. Tests 6 through 13 were performed at differential pressures of between 460 and 1880 psig with valve factors ranging from 0.33 to 0.19. Valve factors appeared to decrease with increasing differential pressure. Another round of differential pressure tests were performed during tests 19 through 25 which were performed at differential pressures between 505 and 1944 psig. These tests were performed at a TSS of 2.0. Valve factors again ranged between 0.28 and 0.17 with valve factors decreasing with increasing differential, pressure. Appendix A3 lists two disk side pressures used for calculating valve factors. The reason for this is that the pressure decreased from its initial value 5-20 psig during unseating. This is believed to be due to the stem being withdrawn from the valve prior to the disk moving. The pressure just before unseating was utilized for calculation of valve factors. Differential pressure tests 46 and 47 were performed by pressurizing the downstream sides of the valve. Both these tests were performed at approximately 1500 psid. Valve factors of 0.15 and 0.16 were obtained.

09/13/95

Bonnet Pressure Response

In accordance with test section E.5 a bonnet depressurization test was performed. The valve was initially set at a TSS of 1.0 to run this test. The bonnet was pressurized by pressurizing the valve with The valve was then closed and the upstream it open. and downstream sides were vented to 0 psig. This first test was started from 900 psig in the bonnet and pressure decreased from approximately 900 to 300 psig over 9 minutes. The next test was performed with a similar setup at a starting pressure of 2000 psig. During this test the pressure decreased from 2000 to 700 psig over 4.25 minutes. Static VOTES test #14 at the same TSS was performed prior to the next depressurization test. Test pressure started at 2000 psig and decreased to 800 psig over 5.25 minutes. The rate is greater at the higher pressure and decreases as pressure decreases. Static VOTES test #15 was performed at a TSS of 2.0 prior to the next depressurization test. Test pressure started at 1980 psig and decreased to 1780 over 5 minutes. During this

testing water could be heard wheezing from the valve seats as pressure decreased. In an effort to ensure that water was leaking past the seats a similar test was performed, however, when the valve was closed to pressurize the bonnet the upstream and downstream sides of the valve were left pressurized such that the upstream and downstream pressures were at bonnet pressure. During this test pressure decreased from 1730 to 1710 psig over 1.75 minutes. This test indicated that a substantial amount of pressure loss was from leakage past the valve seats. Appendix A6 was modified slightly and used to document this testing.

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Pressure lócking testing

Pressure locking data acquisition started with static test 26 at a TSS of 2.0. Each of these pressure lock tests were performed by pressurizing the entire valve with the disk off its seat and then closing the valve and depressurizing the upstream and downstream sides. The bonnet pressure was then recorded with an initial value and an unseating value. Prior to each pressure lock test a static test was performed at the specified test pressure. These were performed because the valve bonnet was pressurized by closing the valve against pressure and then bleeding off the upstream and downstream sides of the valve. VOTES tests 26 through 45 were performed at bonnet pressures between 500 and 2000 psig.

09/15/95

Pressure response to temperature

During this test the valve was closed at a TSS of 2.0. High temperature heat guns were used to heat the valve body and bonnet from the outside. During this test heat guns were directed at both the valve body area and the bonnet area. Temperature of the bonnet water was measured with a contact temperature meter placed on the outside of the bonnet away from the heat guns. The bonnet water temperature is assumed to be approximately equal to the outside bonnet temperature. The test was started at a bonnet pressure of 102 psig and a temperature of 75.8 F. During this test, stem strain was also monitored to verify if stem growth was contributing to the seating force. Time was not monitored for this testing. Appendix A6 was slightly modified and used to document this testing. During this test bonnet pressure initially increased with temperature, then decreased for a short time after which it continued to increase. This test indicated a pressure increase of approximately 2 psig per degree F.

Thermal binding test

The thermal binding test was performed after the valve was heated in accordance with the previous pressure response to temperature test. After heating of the valve, it was opened and closed during VOTES test #48 performed at a TSS of 2.0. Thrust numbers from this test indicated that the sensor had heated and was not providing accurate thrusts. The valve was cooled to ambient conditions and another static test #49 was performed. This test indicated a pullout thrust of 2239 lbs. The increase in pullout was compared to static tests #18, 26 and 27 which were performed at the same TSS. An average pullout value was taken from these three tests of 1906 lbs which yields an increase of 333 lbs. The valve bonnet temperature measured 263 F and the body temperature measured approximately 172 F around the bottom half centerline. The starting valve body and bonnet temperature was 76 F.

Test Summary and Conclusions

Local Leak Rate Testing

Local leak rate testing was performed in accordance with station local leak rate testing methods. The valve was tested from both the upstream and downstream side with each side being tested at two final closing thrusts (TSS of 1.0 and 2.0). This testing indicated zero leakage from both sides of the valve at both torque switch settings. Therefore, the valve was leaktight to within the accuracy of the flow meter or 0.4 scfh.

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Differential Pressure Test

The first set of differential pressure tests were performed from the upstream side of the valve at differential pressures of between 460 and 1880 psid. During analysis of the pressure locking data it was discovered that the seat diameter which was used for calculating open valve factor was the inner seat ring diameter and not the mean seat ring diameter as is used for input into the pressure locking calculations. The actual mean seat diameter was 2.001 inches rather than 3.445. Therefore, using this diameter the valve factors range from 0.13 to 0.21. The second set of differential pressure tests were performed from the upstream side of the valve yielding corrected valve factors in the 0.12 range.

Pressure Locking Test

Pressure locking tests were performed by initially pressurizing the entire valve with the disk off the closed seat. The valve was then closed to trap the pressure in the bonnet and the upstream and downstream seats were vented. Prior to each pressure locking test a static stroke was performed with the entire valve pressurized to the test pressure. The bonnet pressure was then recorded with an initial value and an unseating value. These were performed because the valve bonnet was pressurized by closing the valve against pressure and then bleeding off the upstream and downstream sides of the valve. These static test values were used as input into the pressure locking model. Comparison of the predicted pressure locking forces to the actual tested values are summarized in DOC ID#DG96-000078.

Bonnet Pressure Response Test

The valve was initially closed with a static seating thrust of approximately 13800 lbs. The valve was pressurized to a pressure of approximately 1000 psig and closed and both the upstream and

downstream sides of the valve were vented. When the valve depressurized to 900 psig the test was started. The bonnet depressurized to 300 psig at a decreasing rate over 8.6 minutes. The next test was performed at the same TSS, however, the starting pressure was 2000 psig. During this test the valve depressurized to 700 psig at a decreasing rate over 4.25 minutes. A repeat test was performed starting from 2000 psig. During this test the valve depressurized to 800 psig at a decreasing rate over 5.25 minutes. The valve was then set to a TSS of 2.0 and a measured final thrust of 19870 lbs. The bonnet was pressurized to 1980 psig for an initial pressure. During this test the valve depressurized to 1780 psig at a relatively constant rate over 5.0 minutes. The next test was performed to verify where the leakage was occurring. The valve was pressurized and closed at the same TSS, however, the upstream and downstream sides of the valve were The depressurization rate was substantially slower not vented. starting from 1730 psig depressurizing to 1710 psig over 1.6 minutes. This test indicated that some packing leakage was occurring but this was a small amount of the total. Leakage was not visible through the upper packing but may have been occurring through the capped packing leakoff line. It should be noted that the packing load on this valve was higher than the design value of 1000 lbs per inch of stem diameter. This testing indicates that under bonnet pressurization/depressurization scenarios the seats and packing do leak at a substantial rate even with a valve that measures zero leakage via a LLRT.

Bonnet Pressure Response to Temperature

During this test the bonnet was initially pressurized to 102 psig and the upstream and downstream sides of the valve were depressurized. Heat guns were used to heat the valve and bonnet and a temperature probe was used to measure the bonnet temperature externally. The external bonnet temperature was assumed to approximate the internal fluid temperature. During this test time was not monitored with temperature and pressure due to the length of time to heat the valve. Bonnet pressure started at 102 psig (arbitrary point) and bonnet temperature started at ambient 75.8 F. Initially pressure increased then decreased for a short time with constantly increasing temperature. From this point bonnet pressure increased at an average rate of approximately 2 psig/degree F to a pressure of 201 psig and temperature 263 F. The initial pressure decrease is believed to be due to expansion of the metal and bonnet area due to heating.

Thermal Binding Test

This test was conducted subsequent to heating the valve from the bonnet pressure response to temperature test. Although this test was not part of the procedure it was done to gather additional information. While the valve was at $263 \ F$ it was opened and

closed and then allowed to cool. A VOTES test was performed during this stroke, however, the thrust values were found to be invalid due to heating of the thrust/torque sensor. After cooling of the valve to ambient temperature a VOTES test was performed with a pullout thrust of 2239 lbs. This final test was compared with static tests performed earlier in the testing sequence. The average pullout from three static tests performed at a similar TSS were 1906 lbs with final thrust values from each of these static tests being within 0.48 percent of one another. This is an increase in pullout thrust of 333 lbs or 17.4 percent with a differential temperature of approximately 100 F using the lower valve body temperature as a valve temperature. This increase is much less if the valve bonnet temperature is used.

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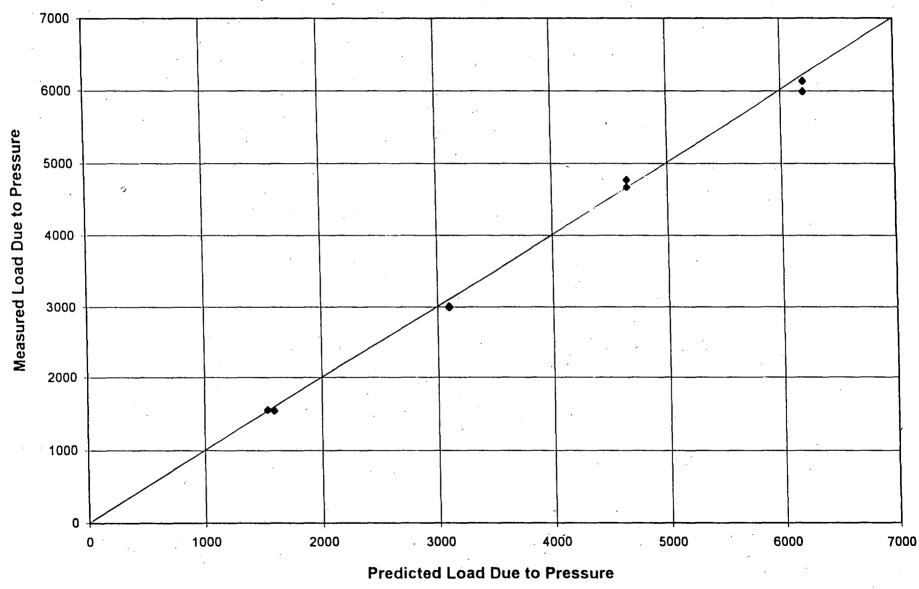
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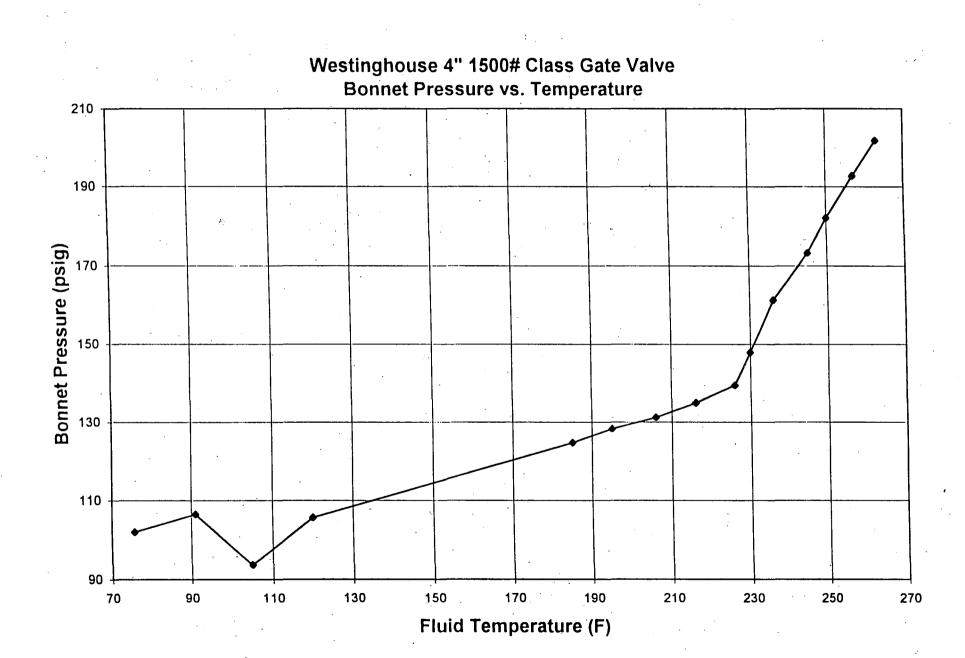
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Westinghouse 4" 1500# Class Gate Valve Measured Pressure Locking Load vs Predicted Load





Attachment 4

ComEd Response to NRC Request for Additional Information on ComEd Pressure Locking Testing