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F	ROM:	Robert L. To	edesco, Assi	stant Direc	tor for Licensing	g, DL	
S	SUBJECT:		ACRS MEETIN 2, JULY 23,		HANNA STEAM ELEC	TRIC STATION,	ł
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TENTATIVE SCHEDULE ACRS SUBCOMMITTEE MEE<u>TING ON SUSOUEHANN</u>A NUCLEAR POWER STATION WASHINGTON, DC THURSDAY, JULY 23, 1981

APPROXIMATE TIME INTRODUCTION 8:30 a.m. Subcommittee Opening Statement Α. W. Kerr, Chairman NRC Staff Introduction Β. 8:35 a.m. R. Stark 1. Overview of OL Review Overview of SER Open Items Pennsylvania Power & Light Co. Introduction 9:00 a.m. С. 1. Site and Plant Description 2. Organization and Management Structure Response to SER Open Items 3. Response to SER Open Items Schedule for Completion of Licensing Review, Sucurity Schedule, 4. Operator Training, Test Program, Fuel Load, and Commercial Operation 10:10 a.m. PENNSYLVANIA POWER & LIGHT COMPANY PRESENTATIONS II. AND NRC STAFF COMMENTS, SEVERAL SPECIFIC ISSUES Management Structure and Technical Resources 10:20 a.m. Α. Compliance with NUREG-0731, etc. 11:00 a.m.. Training and Qualification Program Β. 1. Operator Training and Use of Onsite Simulator 2. Onsite Technical Support Personnel Training 3. Offsite Support Personnel Training Plant Control Room С. '11:40 a.m. 1. Description of Advanced Control Room (ACR) Human Factor Review 2. Control Room Instrumentation (Reg. Guide 1.97 3. and Inadequate Core Cooling Instrumentation) Alternate Shutdown Panel 4.

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JENTATIVE SCHEDULE SUSQUEHANNA

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APPROXIMATE TIME

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	D.	Emergency Planning	12:10	p.m.
Clustart		 Support Facilities Status of Plan (Applicant, County, Staffadt plans, Status of Drill to Test Plan 	FæmA	. § PENA NEVIÈNS)
****	****	**************************************	12:30	p.m.
	E.	Station Electrical Power	1:30	p.m.
Phow		 Loss of AC/Loss of DC (including DC system reliability) 		•
		2. Station Blackout Analysis		
	F.	Decay Heat Removal Capability	2:30	p.m.
Collins		 Normal Mode Degraded Mode 	•	
Slosson	G.	Environment Qualification of Equipment	2:45	p.m.
*****	****	**************************************	3:00	p.m.
Rouse	Η.	Onsite Storage of Spent Fuel and Low-Level Waste, Capacity and Future Plans	3:10	p.m.
Collins Hanne	I. М	Response to NRC Report on Hypothetical BWR Scram System Failures	3:25	p.m.
ł.	J.	Anticipated Transients Without Scram	3:40	p.m.
w. Kennerly		 Plant Protection Measures Operator Training and Procedures Compliance with Propose Rule 		
III.	MARI	K II CONTAINMENT PROGRAM	4:00	p.m.
	Α.	Short-Term Modifications	t	× *
Hanlig	B.'	Long-Term Modifications		4
	с.	Hydrogen Control		

TENTATIVE SCHEDULE SUSQUEHANNA

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SACAES/IV. GEANS	NRC-STAFF-DISCUSSION OF ACRS QUESTIONS ON THE ENVIRONMENTAL IMPACT STATEMENT SUPPLEMENT	4:30 p.m.
۷.	SUSQUEHANNA SECURITY SYSTEM	4:45 p.m.
	(NOTE: Portions of this Session may be Closed as necessary)	
Gaskin	A. Overall Program	
	B. Separation of Units 1 and 2	
VI.	SUBCOMMITTEE DISCUSSION (CAUCUS)	5:00 p.m.
VII.	INSTRUCTIONS TO APPLICANT AND NRC STAFF	5:15 p.m.
ÁDJOL	IRNMENT	5:30 p.m.

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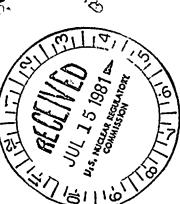
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MEMORANDUM FOR: Walter R. Butler, Chief Containment Systems Branch, DSI

FROM:

SUBJECT: SUSQUEHANNA T-QUENCHER ARM BENDING MOMENT

Mechanical Engineering Branch, DE

Robert J. Bosnak, Chief

REFERENCE: Memorandum from W. Butler to R. Bosnak dated June 10, 1981

JUL: 1 5 1981

In the above referenced memorandum CSB requested that MEB evaluate the applicant's justification for using a 65.1 KNm SRV air clearing bending moment in their design specification rather than the maximum 81.7 KNm bending moment as measured during the Karlstein test. The purpose of this memorandum is to summarize the MEB evaluation of the applicant's justification.

The applicant's justification is based on the assumption that the T-Quencher arm bending moment is used primarily for the evaluation of the T-Quencher body-to-arm.weld. Furthermore, the evaluation of the weld considers additional loadings other than the bending moment. The applicant stated that the additional design specification loadings used in their calculations exceeded the corresponding extrapolated test values and that the overall stress calculation at the body/arm weld using design specification values is conservative.

The applicant has performed calculations using the rules of ASME Code Section III NC-3200*, including Appendix XIII and XIV. The calculations were performed for both the design specification and the extrapolated test data values. A summary of the stresses at the T-Quencher body-to-arm interface point is shown below.

Primary Stress	$(P_{L} + P_{B})$	8107230004 810715 CF ADDCK 05000387
Test Data Design Spec Upset Allowable		CF ADDCR 0000000, CF

*The T-Quencher is an ASME Code Class 3 component. However, the applicant has chosen to use the alternate Class 2 design rules of NC-3200, but has not committed to the additional special requirements of NC-3211.1(d). The Code requires that the special requirements shall be met when NC-3200 rules are used.

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<u>Primary plus Secondary Stresses</u> $(P_1 + P_B + Q)$

Test Data		39.2 ksi
Design Spec	-	33.537 ks1
Upset Allowable	-	42.6 ksi (3 S _m)

As can be seen, the primary (membrane plus bending) stresses due to the total stresses from all associated loadings including pressure and other mechanical loads (e.g., weight, SRV and OBE) is greater for the design specification calculation than for the test data calculation. In addition, the total primary stresses for both the design specification and the test data calculation are less than one-half of the primary stress allowable for the upset condition. Thus, even if a higher bending moment were to be specified in the design specification calculation (i.e., 81.7 KNm vs. 65 KNm), it is not expected that the primary stress allowable would be exceeded.

For primary plus secondary stresses, the test data calculation exceeds the design specification calculation. The specific load combination producing the higher secondary, Q, stresses for the test data calculation is uncertain due to the complexity of the finite element analysis using Appendix XIII rules. In general, the secondary stresses are caused by both thermal stresses and bending stresses at gross structural discontinuities. Local yielding and minor distortions can satisfy the conditions which caused the stresses to occur. Failure is not expected to occur from a single application of the stress. Exceeding the 3 Sm limit on the range of primary plus secondary stresses is acceptable provided a simplified shakedown analysis is performed in accordance with XIII-1153. However, the primary plus secondary stresses for both the design specification and the test data calculations were less than the S Sm elastic limit.

In considering peak stresses, the applicant has performed fatigue evaluations for both design specification and test data values using Appendix XIV rules. The fatigue evaluation conservatively assumed that maximum design specification values would occur for 7000 galve actuations. The design specification bending moment used was 65 KNm. The average test databbending moment measured was 35 KNm, however, in only 3 cases out of 99, the test data bending moment exceeded the 65 KNm design specification bending moment with a maximum measured bending moment of 81.7 KNm. However, the governing stress in the fatigue evaluation was the thermal peak stress and not the mechanical bending moment. The thermal peak stress calculated from the design specification exceeded thermal peak stress calculated from test data values (93.37 ksi vs. 81.9 ksi, respectively). The resulting cumulative usage factor for the design specification calculation exceeded the test data usage factor (0.93 vs. 0.7) even though the test data usage factor was calculated assuming 7000 cycles of the maximum measured bending moment (81.7 KNm). Therefore, it is

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Walter R. Butler

not expected that the larger bending moment from the test data would have a significant effect on the design specification fatigue usage factor and that the relatively few cases where the design specification bending moment is exceeded will not cause fatigue failure.

In summation, for the primary stresses, the design specification calculation resulted in stresses higher than the test data stresses. In both cases the primary stresses were less than one-half the stress allowable. For consideration of primary, secondary and peak stresses, the applicant has performed a conservative fatigue evaluation using maximum design specification values. For the few cases where the bending moment does exceed the design specification value, it is not expected that the higher bending moment will result in fatigue failure.

Therefore, based on our review of the applicant's stress report summary and contingent upon the applicant meeting the special requirements delineated in NR-3211.1(d), we find the Susquehanna load specification for the SRV air clearing bending moment acceptable for use in the design of the T-Quencher body-to-arm weld.

Robert J. Bosnak, Chief Mechanical Engineering Branch Division of Engineering

cc: J. Knight, DE R. Tedesco, DL L. Rubenstein, DSI R. Stark, DL H. Brammer, DE F. Cherny, DE S. Hou, DE J. Kudrick, DSI F. Eltawila, DSI

D. Terao, DE

Contact: D. Terao, MEB:DE X29477

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