



# CALCULATION PACKAGE

FILE/CALC. NO: NUH004.0407

PROJECT NO: NUH-04

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PROJECT NAME:  
NUHOMS 10CFR72 CERTIFICATIONCLIENT:  
PNFSI

## CALCULATION TITLE:

Cask Axial and Radial Thermal Analysis for Standardized NUHOMS®-24P Design With 5-year Old Fuel (1 kW per Fuel Assembly)

## PROBLEM STATEMENT OR OBJECTIVE OF THE CALCULATION:

Determination of the temperature distribution for the NUHOMS®-24P Cask/DSC under several different steady-state design conditions, loaded with 40,000 MWd/MTU, 5-year old fuel.

A 1-dimensional cylindrical model is used to determine the temperature distribution in the Cask/DSC radial and axial directions.

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## 1.0 INTRODUCTION

The objective of the calculation is to determine the standardized NUHOMS®-24P Cask and DSC composite layer temperature distributions in the Cask/DSC axial and radial directions. The value for the decay heat term used in the calculations is based upon 40,000 MWd/MTU, 5-year old PWR fuel (per reference 9).

The calculations include cases for each of the following ambient temperatures: -40° F, 0° F, 70° F, 100° F, and 125° F. Cases are considered to account for the top and bottom effects of the Cask/DSC horizontal orientation (ie., solar load differential and the differing height of the air gap in the annular region between the DSC exterior and the interior Cask cavity). Special cases are considered to determine the maximum DSC shell temperature. Finally, the accident cases are considered in which the Cask neutron shield is considered to be void of material.

Note: The radial cases assume that the Cask outer shell is fabricated from carbon steel in lieu of 304 stainless steel; therefore, the calculations result in inner shell temperatures which are 1 to 2 degrees lower than would result if 304 stainless steel were considered. Although slightly nonconservative, this is considered an insignificant difference.

## 2.0 METHODOLOGY

The methodology used in this calculation is identical to that used in Reference 6. The model and formulas used in the calculation are identical to those used in Reference 4 and are listed in Sections 2.1 and 2.4 of the calculation package.

### 2.1 Description of Radial Model

The cask outside surface temperature is calculated first, and then the temperature distributions across each layer are determined, working inward. Iterations are made for each calculation using temperature dependent material properties. This is repeated until the cask temperature distribution has converged.

All of the temperature drops, including the cask outer surface temperature, are calculated using the thermal resistance concept,

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$$Q = \frac{T_i - T_o}{\Sigma R_n} \quad (1)$$

where Q is the total cask cavity heat load (Btu/hr),  $T_i$  is the inside temperature,  $T_o$  is the outside temperature, and  $R_n$  is the total thermal resistance for heat transfer mode n. Heat transfer modes used are conduction, convection, and radiation. A detailed description of the equations used is given below.

## 2.2 Cask Radial Outer Surface Temperature

The cask outer surface temperature is calculated assuming that heat is transferred from the cask outer surface to the air by convection and radiation. The contribution due to radiation is included in the convection term by superposing their effects as shown below.

$$Q = h_c A (T_o - T_a) + \epsilon \sigma A (T_o^4 - T_a^4) \quad (2)$$

where Q is the total cask cavity heat load (Btu/hr),  $\epsilon$  is the cask outside surface emissivity,  $\sigma$  is the Stefan-Boltzmann constant,  $h_c$  is the convective heat transfer coefficient, and A is the cask surface area for heat transfer. For a horizontal cylinder,  $h_c$  is equal to [4],

$$h_c = 0.18 (T_o - T_a)^{1/3} \quad (3)$$

For a vertical cylinder  $h_c$  is assumed to be that of a vertical plate, and is equal to the value given by Equation 3 after replacing the coefficient 0.18 with 0.19 [4]. The total heat load on the cask outer surface, Q, is the sum of the total cask decay heat and the solar heat flux,

$$Q = Q_{\text{fuel}} + 2\pi r_o l q_{\text{solar}} \quad (4)$$

where  $r_o$  is the cask outside radius, l is the cask inner cavity length,  $Q_{\text{fuel}}$  is the total cask cavity heat load, and  $q_{\text{solar}}$  is the solar heat flux.

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By defining a radiation coefficient  $h_r$  as,

$$h_r = \epsilon \sigma (T_o + T_a) (T_o^2 + T_a^2) \quad (5)$$

Equation 2 can be rearranged into the form,

$$T_o = \frac{Q}{A(h_c + h_r)} + T_a \quad (6)$$

This relation gives the outside cask temperature in terms of the total cask heat load, ambient temperature, cask area, and the sum of the heat transfer coefficients. Because the heat transfer coefficients are in turn dependent on the cask surface temperature, iterations are necessary to determine the cask surface temperature.

### 2.3 Radial Internal Temperature Drops

The only mode of heat transfer assumed to occur in the internal cask layers is conduction. The thermal resistance for conduction across concentric cylinders is equal to,

$$R_c = \frac{\ln \left( \frac{r_o}{r_i} \right)}{2\pi k l} \quad (7)$$

where  $r_o$  is the outside radius,  $r_i$  is the inside radius, and  $k$  is the thermal conductivity of the layer material. Substituting into Equation 1 and rearranging gives,

$$T_i = \frac{Q \cdot \ln \left( \frac{r_o}{r_i} \right)}{2\pi k l} + T_o \quad (8)$$

where  $Q$  in this case is equal to the total decay heat generation rate of the fuel assemblies.

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## 2.3.1 Radial Internal Thermal Radiation

Layers through which there is thermal radiation heat transfer are modelled as having two thermal resistances in parallel, radiation and conduction. The total thermal resistance for two resistances in parallel is given by,

$$R_{\text{total}} = \frac{1}{\frac{1}{R_c} + \frac{1}{R_r}} \quad (9)$$

The resistance due to conduction,  $R_c$  is identical to that in Equation 7, and the resistance due to radiation is given by,

$$R_r = \frac{1}{2\pi h_r r_i l} \quad (10)$$

where the radiation coefficient  $h_r$  for two concentric cylinders is given by,

$$h_r = \frac{\sigma (T_i + T_o) (T_i^2 + T_o^2)}{\frac{1}{\epsilon_i} + \frac{r_i}{r_o} \left[ \frac{1}{\epsilon_o} - 1 \right]} \quad (11)$$

where  $\epsilon_i$  and  $\epsilon_o$  are the emissivities of the inside and outside surfaces respectively. Substituting Equations 7, 9, and 10 into Equation 1 and rearranging gives the temperature on the inside surface,

$$T_i = \frac{Q}{2\pi h_r r_i l + \frac{2\pi k l}{\ln \left[ \frac{r_o}{r_i} \right]}} + T_o \quad (12)$$

The radiation coefficient  $h_r$  is calculated by assuming temperatures for Equation 11 and iterating till convergence is achieved.

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## 2.3.2 Radial Internal Convection

Internal convection in an annular region is modeled by substituting the thermal conductivity with an effective thermal conductivity which includes the effects of convection. Effective conductivities are provided in the material library for air and water (airconv and h20conv). Credit for convection in an annular region is taken if the Grashof number for that annular region falls within the turbulent regime, ie.:

$$Gr_g = \frac{g\beta(T_i - T_o)(r_o - r_i)^3}{\nu^2} \quad (13)$$

where  $\beta$  is the thermal coefficient of volume expansion,  $g$  is the gravitational constant, and  $\nu$  is the kinematic viscosity. The fluid flow in the annulus must be in the turbulent regime in order for the convection corrected conductivities to be used. Turbulent flow occurs when the product of the Grashof and the Prandtl numbers ( $Gr_g Pr$ ) is greater than  $10^7$  [2].

## 2.4 Description of Axial Thermal Model

The methodology used for the axial case is identical to the technique employed in the cask radial thermal analysis (refer to Section 2.1). The difference between the radial and axial cases is in the equations used to calculate the heat loads and the thermal resistances. These differences are discussed below.

### 2.4.1 Axial Heat Fluxes

For the axial case, all layers are assumed to be flat plates, so heat fluxes are not dependent on the axial location. Therefore, a unit area heat flux,  $q$  (Btu/hr·ft<sup>2</sup>), is used in the axial analysis. The heat flux in the cask ends due to the fuel decay heat is chosen to be equal to the decay heat divided by the entire cask internal cavity area [5]:

$$q_{fuel} = \frac{Q_{fuel}}{2\pi r_i^2 + 2\pi r_i l} \quad (14)$$

The solar heat flux on the cask outside ends is set equal to 20% of the input solar heat flux for a horizontal cask [11], to 100% of the input solar heat flux for the top end of a vertical cask, and to zero for the bottom end of a vertical cask.

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## 2.4.2 Cask Axial Outer Surface Temperature

For the bottom end of a vertical cask, the surface temperature is set equal to the ambient temperature (representing the floor). The unit area heat flux is equal to  $Q/A_{\text{cask}}$ , and the equation for determining the cask surface temperature for the other orientations is:

$$T_o = \frac{q}{h_r + h_c} + T_a \quad (15)$$

The convective heat transfer coefficient,  $h_c$ , for a vertical flat plate is given by [5]:

$$h_c = 0.19 (T_o - T_a)^{\frac{1}{3}} \quad (16)$$

The radiation heat transfer coefficient,  $h_r$ , for a vertical flat plate is given by [5]:

$$h_r = \epsilon \cdot \sigma \cdot (T_o + T_a) \cdot (T_o^2 + T_a^2) \quad (17)$$

## 2.4.3 Axial Conduction

For conduction across an internal layer, the inside layer temperature is given by,

$$T_i = \frac{q}{k} (L_o - L_i) + T_o \quad (18)$$

Where  $L_o - L_i$  is the thickness of the layer.

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## 2.4.4 Axial Internal Radiation

The radiation coefficient for radiation between two parallel flat plates is given by,

$$h_r = \frac{\sigma (T_i + T_o) (T_i^2 + T_o^2)}{\frac{1}{\epsilon_i} + \frac{1}{\epsilon_o} - 1} \quad (19)$$

The inside temperature is given by

$$T_i = \frac{q}{h_r + \frac{k}{L_o - L_i}} + T_o \quad (20)$$

or, rearranging the terms yields equation (8) below:

$$q = (T_i - T_o) \times \left[ h_r + \frac{k}{L_o - L_i} \right] \quad (21)$$

## 2.5 Cask/DSC Radial Geometry

The thermal calculations herein are based upon the Cask and DSC shell lengths and radii specified in Section 3.0.

An interior Cask cavity length of 187.75" is assumed to form the basis for the Cask shell temperature determinations (assuming the cask cover plate is not in place). With the cask cover plate the cask cavity is 196.75". If 196.75" were used as the cask cavity length in the calculations, the cask shell temperatures would increase by approximately one degree Fahrenheit. This would be a negligible increase in cask shell temperatures.

When the maximum DSC outer shell temperature is calculated, to ensure conservative results, the DSC interior cavity length of 167" is assumed to be exposed to the radial decay heat.

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## 2.6 Cask/DSC Axial Geometry

The thermal calculations herein are based upon the Cask and DSC shell lengths and radii specified in Section 3.0.

An interior Cask cavity length of 187.75" is assumed to form the basis for the Cask axial shell temperature determinations (assuming the cask cover plate is not in place). With the cask cover plate installed, the cask cavity is 196.75". If 196.75" were used as the cask cavity length in the calculations, the cask axial shell temperatures would increase by approximately one degree Fahrenheit. This would be a negligible increase in cask shell temperatures.

When the maximum DSC outer shell temperature is calculated, to ensure conservative results, the DSC interior cavity length of 167" is used to calculate the heat flux through the DSC surfaces.

A 0.50" air gap was conservatively assumed for the maximum DSC exterior temperature horizontal orientation calculations. The actual (70° F) longitudinal clearance between the DSC and cask is 0.75" under cold conditions; however, to determine a conservative heating effect on the DSC, it is assumed that a 0.50" air gap exists at both the top and bottom ends. The use of a 0.5" air gap at the top and bottom ends of the DSC is conservative; the thermal expansion of the DSC (due to fuel assembly decay heat) will reduce the gap below 0.50".

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## 2.7 Material Properties

The thermal conductivities of the materials in the Cask and DSC are listed below:

### 304 Stainless Steel (Reference 5)

T (°F)	k (Btu/hr-ft <sup>2</sup> -°F)
32	8.1
212	8.7
392	8.7
572	9.4
752	10
1112	11
1472	13

### Carbon Steel (Reference 5)

T (°F)	k (Btu/hr-ft <sup>2</sup> -°F)
32	25
212	25
392	24
572	23
752	21
1112	19
1472	17
1832	16

### Lead (Reference 5)

T (°F)	k (Btu/hr-ft <sup>2</sup> -°F)
32	20.3
212	19.3
392	18.2
572	17.2
630	12.1
717	9.7
800	9
980	8.7
1276	8.6

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**Air (Reference 5)**

T (°F)	k (@ 14.7 psia) (Btu/hr-ft <sup>2</sup> -°F)
80.6	0.0151
170	0.0172
261	0.0191
351	0.021
441	0.0228
531	0.0246
621	0.0263
711	0.028

**Airconv (Reference 5)**

(Air with natural convection)

T (°F)	k (@ 14.7 psia) (Btu/hr-ft <sup>2</sup> -°F)
150	0.1

**NS-3 with Stiffeners**

T (°F)	k @ 250° F (Btu/hr-ft <sup>2</sup> -°F)
150	1.13 (refer to Section 3.4 for calculation)

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### 3.0 CALCULATIONS

#### 3.1 Radial Thermal Cases

This section includes the results of the calculations (which utilizes the methodology described in Section 2). The results include the input data and the Cask/DSC temperature distribution output data for the different cases. The following radial thermal cases were considered:

Case <sup>1</sup>	Orientation/Top or Bottom Model	Ambient	Solar Yes/No	Solar Load (btuh/ft <sup>2</sup> ) [1]	Remarks
1A	Horizontal/Top	-40	No	0	0.375" Air Gap
1B	Horizontal/Bottom	-40	No	0	No Air Gap
1C	Horizontal	-40	No	0	0.75" Air Gap w/ Cask Cav Length of 167"
2A	Horizontal/Top	0	No	0	0.375" Air Gap
2B	Horizontal/Bottom	0	No	0	No Air Gap
2C	Horizontal	0	No	0	0.75" Air Gap w/ Cask Cav Length of 167"
3A	Horizontal/Top	70 <sup>1</sup>	Yes	62	0.375" Air Gap
3B	Horizontal/Bottom	70	No	0	No Air Gap
3C	Horizontal	70	Yes	62	0.75" Air Gap w/ Cask Cav Length of 167"
3D	Vertical	70	No	0	0.375" Air Gap
4A	Horizontal/Top	100	Yes	62	0.375" Air Gap
4B	Horizontal/Bottom	100	No	0	No Air Gap
4C	Horizontal	100	Yes	62	0.75" Air Gap w/ Cask Cav Length of 167"
4D	Vertical	100	No	0	
5A	Horizontal/Top	125	Yes	123	0.375" Air Gap
5B	Horizontal/Bottom	125	No	0	No Air Gap

<sup>1</sup> Cases 1C, 2C, 3C, 4C and 5C are used for determining the maximum DSC surface temperature - only.

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5C	Horizontal	125	Yes	123	0.75" Air Gap w/ Cask Cav Length of 167"
5D	Horizontal/Accid ent	125	Yes	123	0.375" Air Gap w/ no neutron shield
5E	Horizontal/Accid ent	125	Yes	123	0.75" Air Gap w/ no neu sh & 167" cav

### 3.2 Axial Thermal Cases

This section includes the results of the calculations (which utilizes the methodology described in Section 2). The results include the input data and the Cask/DSC temperature distribution output data for the different cases. The following axial thermal cases were considered:

Case <sup>2</sup>	Orientation/Top or Bottom Model	Ambient Temp (F)	Solar Yes/No	Solar Load (btuh/ft <sup>2</sup> )	Remarks
1A	Horizontal/Top	-40	No	0	0.5" Air Gap & 167" Cask Cavity
1B	Horizontal/Top	-40	No	0	No Air Gap
2A	Horizontal/Top	0	No	0	0.5" Air Gap & 167" Cask Cavity
2B	Horizontal/Top	0	No	0	No Air Gap
3A	Horizontal/Top	70	Yes	62	0.5" Air Gap & 167" Cask Cavity
3B	Horizontal/Top	70	Yes	62	No Air Gap
4A	Horizontal/Top	100	Yes	62	0.5" Air Gap & 167" Cask Cavity
4B	Horizontal/Top	100	Yes	62	No Air Gap
4C	Vertical	100	No	0	
5A	Horizontal/Top	125	Yes	123	0.5" Air Gap & 167" Cask Cavity
5B	Horizontal/Top	125	Yes	123	No Air Gap
6A	Horizontal/Botto m	-40	No	0	0.5" Air Gap & 167" Cask Cavity

<sup>2</sup> Cases 1A, 2A, 3A, 4A, 5A, 6A, 7A, 8A, 9A & 10A were included for determining the DSC axial temperature distributions - only.

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6B	Horizontal/Bottom	-40	No	0	No Air Gap
7A	Horizontal/Bottom	0	No	0	0.5" Air Gap & 167" Cask Cavity
7B	Horizontal/Bottom	0	No	0	No Air Gap
8A	Horizontal/Bottom	70	Yes	62	0.5" Air Gap & 167" Cask Cavity
8B	Horizontal/Bottom	70	Yes	62	No Air Gap
9A	Horizontal/Bottom	100	Yes	62	0.5" Air Gap & 167" Cask Cavity
9B	Horizontal/Bottom	100	Yes	62	No Air Gap
9C	Vertical	100	No	0	
10A	Horizontal/Bottom	125	Yes	123	0.5" Air Gap & 167" Cask Cavity
10B	Horizontal/Bottom	125	Yes	123	No Air Gap

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### 3.3 Fuel Decay Heat Calculation

The unit fuel decay heat is based upon PWR fuel with a 30,000 MWd/MTU burnup, 5-year old fuel. The decay heat rate for this fuel is given as:

1.0 kW per fuel assembly (reference 9)

The temperature distributions calculated herein are valid for all PWR fuel types and various cooling times as long as the decay heat per fuel assembly is less than or equal to 1.0 kW. The heat flux used herein is based upon the above decay heat rate per fuel assembly, the number of assemblies per DSC, and the ratio of the radial surface area to total surface area of the DSC.

The radial heat is calculated as follows:

$$Q = \frac{(1kW) \times (24f/a) \times \pi \times D \times L}{\pi \times D \times L + \pi \times \frac{D^2}{2}}$$

Which reduces to:

$$Q = \frac{(1kW) \times (24f/a) \times \pi \times L}{\pi \times L + \pi \times \frac{D}{2}}$$

The heat flux used for the axial case is calculated based upon the total decay heat divided by the interior surface area of the cask.

Where:

- Q = Decay heat (kW)
- L = Cask cavity length
- D = Cask interior radius

$$Q = \frac{(1.0kW) \times (24f/a) \times 144}{2 \times \pi \times r \times L + 2 \times \pi \times r^2}$$

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Note: Per the sample problem presented in Section 3.5.2, the axial heat flux used is 0.08 kW/ft<sup>2</sup> or 274.8 Btu/hr/ft<sup>2</sup>.

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### 3.4 Calculation of NS-3-Stiff Thermal Conductivity

The calculation of the thermal conductivity of the neutron shield is calculated using a weighted average of Bisco NS-3 and stainless steel (ie., taking into account the twenty four 304 stainless steel support angle strips).

The mass of steel in the neutron shield is calculated by (refer to Ref 6):

Vol of steel = 0.125" x 4.25" x 177.85"

Vol of steel = 94.48 in<sup>3</sup> per strip

Mass of steel (total) = 94.48 in<sup>3</sup> \* 24 strips \* 492 lbs/ft<sup>3</sup> \* 1/1728

Mass of steel (total) = 645.7 lbm

The mass of NS-3 is calculated by:

Vol of NS-3 =  $\{(\pi/4) * (85^2 - 79^2)(176.85) - (94.48)(24)\}/1728$

Vol of NS-3 = 77.8 ft<sup>3</sup>

Mass of NS-3 = 77.8 ft<sup>3</sup> \* 110 lbs/ft<sup>3</sup> (Reference 6)

Mass of NS-3 = 8556 lbm

The average value of the thermal conductivity,  $k_{NS3-stif}$ , is determined by:

$$k_{NS3-stif} = (8556 \text{ lbm})(k_{ns3}) + (645.7 \text{ lbm})(k_{304ss}) / (8556 + 645.7)$$

Where:  $k_{ns3} = 0.4883 \text{ Btu/hr/ft}^2\text{°F @ } 150\text{° F}$  (reference 6)

$k_{304ss} = 9.0 \text{ Btu/hr/ft}^2\text{°F @ } 150\text{°F}$  (see thermal conductivities on page 11 & note below)

$k_{304ss} = 9.6 \text{ Btu/hr/ft}^2\text{°F @ } 250\text{°F}$  (see page 11 & note below)

Substituting yields (see note below):

$$k_{NS3-stif} = (8556 \text{ lbm})(0.4883) + (645.7 \text{ lbm})(9.6) / (8556 + 645.7)$$

$$k_{NS3-stif} = 1.127 \text{ say } 1.13 \text{ Btu/hr/ft}^2\text{°F}$$

Note: Accident cases 5D & 5E described in Section 3.1, 3.2 are the most conservative cases considered and therefore govern the design temperature at which the thermal conductivity of NS3-stif is evaluated. These cases will yield temperatures in the neutron shield in excess of 250° F, therefore, the thermal conductivity of 1.13 is used. All other cases use the same constant value for  $k_{NS3-stif}$ ; with a minimal non-conservative effect in the calculated shell temperatures for the -40° F, 0° F and

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70° F cases directly related to the temperature dependent thermal conductivity of the 304 stainless steel stiffeners.

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## 3.5 Sample Problems

### 3.5.1 Radial Case Sample Problem

The following is a sample problem which demonstrates the use of the Section 2 methodology for the radial case (Radial Case 5E):

Outside Temperature:

$$Q = (1 * 24.000) * 3414 * (167.00 / (167.00 + 33.00)) + (2 * 3.1416 * (42.63/12) * (167.00/12) * 123.0)$$
$$Q = 106624 \text{ Btu/hr}$$
$$A = (2 * 3.1416 * 42.63 * 167.00 / 144) = 310.6 \text{ ft}^2$$
$$hc = 0.18 * (282.7 - 125.0)^{1/3} = 0.9726 \text{ Btu/hr} \cdot \text{ft}^2 \cdot \text{°F}$$
$$hr = (0.587 * 1.73E-09 * (282.7 + 125.0 + 919.34) * ((282.7 + 459.67)^2 + (125.0 + 459.67)^2))$$
$$hr = 1.2034 \text{ Btu/hr} \cdot \text{ft}^2 \cdot \text{°F}$$

$$T_{out} = (106624 / (310.6 * (0.9726 + 1.2034)) + 125.0) = 282.7 \text{ °F}$$

Agrees with initial estimate of 282.7 °F

Inside Temperatures

$$Q = (1 * 24.000) * 3414 * (167.00 / (167.00 + 33.00)) = 68416.6 \text{ Btu/hr}$$

Layer: N. Shield Panel

$$T_{avg} = (283.0 + 282.7) / 2 = 282.9 \text{ °F}$$

k of ss304 at 282.9 °F is given by: [5]

$$k = (282.9 - 250.0) * (9.8000 - 9.6000) / (300.0 - 250.0) + 9.6000$$
$$= 9.7315 \text{ Btu/hr} \cdot \text{ft} \cdot \text{°F}$$

No thermal radiation in layer

The conduction term is given by:

$$(2 * 3.1416 * 9.7315 * (167.00 / 12)) / \text{LN}(42.63 / 42.50) = 289741.117 \text{ Btu/hr} \cdot \text{°F}$$

$$T_i = (68417) / (2 * 3.1416 * (42.50 / 12) * (167.00 / 12) * 0.000 + 289741.1) + 282.7 = 283.0 \text{ °F}$$

Agrees with initial estimate of 283.0 °F

Layer: Neutron Shield

$$T_{avg} = (405.0 + 283.0) / 2 = 344.0 \text{ °F}$$

k of airconv at 344.0 °F is given by: [5]

$$k = 0.1000 \text{ Btu/hr} \cdot \text{ft} \cdot \text{°F}$$

$$hr = (1.73E-09 * (405.0 + 283.0 + 919.34) * ((405.0 + 459.67)^2 + (283.0 + 459.67)^2)) / (1/0.587 + (39.50/42.50) * (1/0.587 - 1)) = 1.533 \text{ Btu/hr} \cdot \text{ft}^2 \cdot \text{°F}$$

The conduction term is given by:

$$(2 * 3.1416 * 0.1000 * (167.00 / 12)) / \text{LN}(42.50 / 39.50) = 119.449 \text{ Btu/hr} \cdot \text{°F}$$

$$T_i = (68417) / (2 * 3.1416 * (39.50 / 12) * (167.00 / 12) * 1.533 + 119.4) + 283.0 = 405.0 \text{ °F}$$

Agrees with initial estimate of 405.0 °F

Layer: Outer Shell

$$T_{avg} = (406.3 + 405.0) / 2 = 405.7 \text{ °F}$$

k of csteel at 405.7 °F is given by: [5]

$$k = (405.7 - 392.0) * (23.0000 - 24.0000) / (572.0 - 392.0) + 24.0000$$
$$= 23.9241 \text{ Btu/hr} \cdot \text{ft} \cdot \text{°F}$$

No thermal radiation in layer

The conduction term is given by:

$$(2 * 3.1416 * 23.9241 * (167.00 / 12)) / \text{LN}(39.50 / 38.00) = 54035.111 \text{ Btu/hr} \cdot \text{°F}$$

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$T_i = (68417)/(2*3.1416*(38.00/12)*(167.00/12)*0.000+54035.1)+405.0=406.3 \text{ } ^\circ\text{F}$   
Agrees with initial estimate of 406.3 °F

Layer: Gamma Shield

$T_{avg} = (410.5+406.3)/2 = 408.4 \text{ } ^\circ\text{F}$

k of lead at 408.4 °F is given by: [5]

$k = (408.4-392.0)*(17.2000-18.2000)/(572.0-392.0)+18.2000$   
 $= 18.1089 \text{ Btu/hr}\cdot\text{ft}\cdot^\circ\text{F}$

No thermal radiation in layer

The conduction term is given by:

$(2*3.1416*18.1089*(167.00/12))/LN(38.00/34.50)=16387.410 \text{ Btu/hr}\cdot^\circ\text{F}$

$T_i = (68417)/(2*3.1416*(34.50/12)*(167.00/12)*0.000+16387.4)+406.3=410.5 \text{ } ^\circ\text{F}$   
Agrees with initial estimate of 410.5 °F

Layer: Inner Shell

$T_{avg} = (411.6+410.5)/2 = 411.0 \text{ } ^\circ\text{F}$

k of ss304 at 411.0 °F is given by: [5]

$k = (411.0-400.0)*(10.6000-10.4000)/(450.0-400.0)+10.4000$   
 $= 10.4441 \text{ Btu/hr}\cdot\text{ft}\cdot^\circ\text{F}$

No thermal radiation in layer

The conduction term is given by:

$(2*3.1416*10.4441*(167.00/12))/LN(34.50/34.00)=62556.011 \text{ Btu/hr}\cdot^\circ\text{F}$

$T_i = (68417)/(2*3.1416*(34.00/12)*(167.00/12)*0.000+62556.0)+410.5=411.6 \text{ } ^\circ\text{F}$   
Agrees with initial estimate of 411.6 °F

Layer: Air Gap

$T_{avg} = (517.7+411.6)/2 = 464.6 \text{ } ^\circ\text{F}$

k of air at 464.6 °F is given by: [5]

$k = (464.6-392.0)*(0.0248-0.0214)/(572.0-392.0)+0.0214$   
 $= 0.0228 \text{ Btu/hr}\cdot\text{ft}\cdot^\circ\text{F}$

$hr = (1.73E-09*(517.7+411.6+919.34)*((517.7+459.67)^2+(411.6+459.67)^2))/$   
 $(1/0.587+(33.25/34.00)*(1/0.587-1))=2.292 \text{ Btu/hr}\cdot\text{ft}^2\cdot^\circ\text{F}$

The conduction term is given by:

$(2*3.1416*0.0228*(167.00/12))/LN(34.00/33.25)=89.269 \text{ Btu/hr}\cdot^\circ\text{F}$

$T_i = (68417)/(2*3.1416*(33.25/12)*(167.00/12)*2.292+89.3)+411.6=517.7 \text{ } ^\circ\text{F}$   
Agrees with initial estimate of 517.7 °F

Layer: DSC

$T_{avg} = (518.2+517.7)/2 = 518.0 \text{ } ^\circ\text{F}$

k of ss304 at 518.0 °F is given by: [5]

$k = (518.0-500.0)*(11.1000-10.9000)/(550.0-500.0)+10.9000$   
 $= 10.9719 \text{ Btu/hr}\cdot\text{ft}\cdot^\circ\text{F}$

No thermal radiation in layer

The conduction term is given by:

$(2*3.1416*10.9719*(167.00/12))/LN(33.25/33.00)=127118.707 \text{ Btu/hr}\cdot^\circ\text{F}$

$T_i = (68417)/(2*3.1416*(33.00/12)*(167.00/12)*0.000+127118.7)+517.7=518.2 \text{ } ^\circ\text{F}$   
Agrees with initial estimate of 518.2 °F

All layers agree with initial guesses.  
Calculations are completed.

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## 3.5.2 Axial Case Sample Problem

The following is a sample problem which demonstrates the use of the Section 2 methodology for the axial case:

Case 10A: 125 F Ambient, Top End, W/ Solar, W/ Air Gap, 167" Cask Cavity.

### Orientation:

Emissivity of Cask Outer Surface = 0.587  
Ambient Temperature = 125 inches  
Solar Heat Load = 123 Btu/hr/ft<sup>2</sup>  
Cask Internal Radius = 34 inches  
Cask Cavity Length = 167 inches

### Heat Flux Calculations:

Decay Heat of Fuel Assembly = 1.00 kW  
Decay Heat (per DSC w/ 24 f/a's), Q = 24.0 kW  
Decay Heat (per DSC w/ 24 f/a's), Q = 81946 Btu/hr  
Vertical Surface Solar Heat Flux = 0.2 \* 123 = 25 Btu/hr  
Unit Decay Heat, q = 81946 / (2 \* Pi \* 34 + 2 \* Pi \* 34 \* 167) \* 144 = 274.8 Btu/hr/ft<sup>2</sup>  
Total Exterior Vert Surf Heat Flux = 24.6 + 274.8 = 299.4 Btu/hr/ft<sup>2</sup>

### Layer: Exterior Plate

Assume Delta T = 140.5 F  
Tamb = 125.000 F  
Tamb = 585.000 R  
To = 265.523 F  
To = 725.193 R  
hc = 0.19 \* (140.523)<sup>1/3</sup> = 0.988  
hr = 0.587 \* (585.0 + 725.2) \* (585.0<sup>2</sup> + 725.2<sup>2</sup>) \* (0.1714E-8)  
= 1.135 Btu/hr/ft<sup>2</sup>/F  
q = 140.523 \* (0.988 + 1.135) = 298 Btu/hr/ft<sup>2</sup>

### Layer: Bottom Plate

To = 265.523 F  
Average Plate Temperature = 266.839 F  
k @ 266.839 F = 8.700 Btu/hr/ft/F  
Plate thickness = 1.00 inches  
Ti = 275 / 8.70 \* (1.00 / 12) + 265.52 = 268.155 F  
Delta T = 268.155 - 265.523 = 2.632 F

### Layer: Neutron Shield

To = 268.155 F  
Average Plate Temperature = 320.928 F  
k @ 320.928 F = 0.488 Btu/hr/ft/F  
Plate thickness = 2.25 inches  
Ti = 275 / 0.488 \* (2.25 / 12) + 268.15 = 373.701 F  
Delta T = 373.701 - 268.155 = 105.546 F

### Bottom Struc Plate Temperature Calculations:

To = 373.701 F  
Average Plate Temperature = 376.004 F  
k @ 376.004 F = 8.700 Btu/hr/ft/F  
Plate thickness = 1.75 inches

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$$T_i = 275/8.70 * (1.75/12) + 373.70 = 378.307 \text{ F}$$
$$\Delta T = 378.307 - 373.701 = 4.607 \text{ F}$$

Layer: Air Gap

$$T_o = 378.307 \text{ F}$$

$$T_o = 837.977 \text{ R}$$

$$\text{Assumed } T_i = 485.921 \text{ F}$$

$$\text{Assumed } T_i = 945.591 \text{ R}$$

$$\text{Resulting Assumed } \Delta T = 107.614 \text{ R}$$

$$\text{Air Gap Thickness} = 0.50 \text{ inches}$$

$$E_o = 0.587$$

$$E_i = 0.587$$

$$1/E_o = 1.704$$

$$1/E_i = 1.704$$

$$(1/E_i + 1/E_o) - 1 = 2.407$$

$$hr = (0.17E-8) * (837.977 + 945.591) * (837.977^2 + 945.591^2) / 2.407$$
$$= 2.011 \text{ Btu/hr/ft}^2/\text{F}$$

$$k = 0.023 \text{ Btu/hr/ft/F}$$

$$k/0.50/12 = 0.543$$

$$\Delta T (\text{incl conduction}) = 275 / (2.011 + 0.023) = 107.614 \text{ F}$$

$$T_i (\text{calc'd - including conduction}) = 107.614 + 378.307 = 485.921 \text{ F}$$

Layer: DSC Top Plate

$$T_o = 485.921 \text{ F}$$

$$\text{Average Plate Temperature} = 488.129 \text{ F}$$

$$k @ 488.129 \text{ F} = 9.074 \text{ Btu/hr/ft/F}$$

$$\text{Plate thickness} = 1.75 \text{ inches}$$

$$T_i = 275/9.074 * (1.75/12) + 485.92 = 490.337 \text{ F}$$

Layer: DSC Plug

$$T_o = 490.337 \text{ F}$$

$$\text{Average Plate Temperature} = 499.129 \text{ F}$$

$$k @ 499.129 \text{ F} = 9.117 \text{ Btu/hr/ft/F}$$

$$\text{Plate thickness} = 7.00 \text{ inches}$$

$$T_i = 275/9.117 * (7.00/12) + 490.337 = 507.921 \text{ F}$$

All temperatures agree with initial guesses.

The axial layer sample problem is complete.

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### 3.6 Results for Cask/DSC Radial Cases

Case 1A: -40° F, No solar, Horizontal w/ 0.375" air gap  
 (Cask top model)

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature -40 °F  
 Solar Heat Load 0 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surface 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 187.75 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr·ft·°F)	k	Ti (°F)	To (°F)
<i>DSC</i>	<i>ss304</i>	<i>0.625</i>	<i>no</i>	<i>9.92E+00</i>	<i>321.3</i>	<i>320.0</i>
<i>Air Gap</i>	<i>air</i>	<i>0.375</i>	<i>yes</i>	<i>1.83E-02</i>	<i>320.0</i>	<i>161.0</i>
Inner Shell	ss304	0.500	no	9.06E+00	161.0	159.8
Gamma Shield	lead	3.500	no	1.96E+01	159.8	156.3
Outer Shell	csteel	1.500	no	2.50E+01	156.3	155.2
Neutron Shield	ns3stif	3.000	no	9.49E-01	155.2	100.5
N. Shield Panel	ss304	0.125	no	8.70E+00	100.5	100.3

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Case 1B: -40° F, No solar, Horizontal w/ no air gap  
 (Cask bottom model)

Number of Assemblies	24
Decay Heat per Assembly	1 kW
Ambient Temperature	-40 °F
Solar Heat Load	0 Btu/hr-ft <sup>2</sup>
Emissivity of Outer Surfac	0.587
Cask Inner Radius	33 in
Cask Cavity Length	187.75 in
Orientation	Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr-ft·°F)	k	Ti (°F)	To (°F)
<i>DSC</i>	<i>ss304</i>	<i>0.625</i>	<i>no</i>	<i>9.08E+00</i>	<i>163.9</i>	<i>162.4</i>
<i>Air Gap</i>	<i>air</i>	<i>0.000</i>	<i>no</i>	<i>1.66E-02</i>	<i>162.4</i>	<i>162.4</i>
Inner Shell	ss304	0.500	no	9.07E+00	162.4	161.3
Gamma Shield	lead	3.500	no	1.96E+01	161.3	157.7
Outer Shell	csteel	1.500	no	2.50E+01	157.7	156.6
Neutron Shield	ns3stif	3.000	no	9.49E-01	156.6	101.4
N. Shield Panel	ss304	0.125	no	8.71E+00	101.4	101.2

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Case 1C: -40° F, No solar, Horizontal w/ 0.75" air gap  
 (Model to determine maximum DSC shell temperature only)

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature -40 °F  
 Solar Heat Load 0 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surface 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 167 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr·ft·°F)	k	Ti (°F)	To (°F)
DSC	ss304	0.250	no	1.02E+01	370.5	370.0
Air Gap	air	0.750	yes	1.90E-02	370.0	177.6
<i>Inner Shell</i>	<i>ss304</i>	<i>0.500</i>	<i>no</i>	<i>9.16E+00</i>	<i>177.6</i>	<i>176.4</i>
<i>Gamma Shield</i>	<i>lead</i>	<i>3.500</i>	<i>no</i>	<i>1.95E+01</i>	<i>176.4</i>	<i>172.5</i>
<i>Outer Shell</i>	<i>csteel</i>	<i>1.500</i>	<i>no</i>	<i>2.50E+01</i>	<i>172.5</i>	<i>171.3</i>
<i>Neutron Shield</i>	<i>ns3stif</i>	<i>3.000</i>	<i>no</i>	<i>9.49E-01</i>	<i>171.3</i>	<i>110.9</i>
<i>N. Shield Panel</i>	<i>ss304</i>	<i>0.125</i>	<i>no</i>	<i>8.76E+00</i>	<i>110.9</i>	<i>110.7</i>

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Case 2A: 0° F, No solar, Horizontal w/ 0.375" air gap  
 (Cask top model)

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 0 °F  
 Solar Heat Load 0 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surface 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 187.75 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr·ft·°F)	k	Ti (°F)	To (°F)
<i>DSC</i>	<i>ss304</i>	<i>0.625</i>	<i>no</i>	<i>1.00E+01</i>	<i>340.9</i>	<i>339.6</i>
<i>Air Gap</i>	<i>air</i>	<i>0.375</i>	<i>yes</i>	<i>1.88E-02</i>	<i>339.6</i>	<i>192.4</i>
Inner Shell	ss304	0.500	no	9.25E+00	192.4	191.3
Gamma Shield	lead	3.500	no	1.94E+01	191.3	187.8
Outer Shell	csteel	1.500	no	2.50E+01	187.8	186.7
Neutron Shield	ns3stif	3.000	no	9.49E-01	186.7	132.0
N. Shield Panel	ss304	0.125	no	8.89E+00	132.0	131.7

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Case 2B: 0° F, No solar, Horizontal w/ no air gap  
 (Cask bottom model)

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 0 °F  
 Solar Heat Load 0 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surfac 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 187.75 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr·ft·°F)	k	Ti (°F)	To (°F)
<i>DSC</i>	<i>ss304</i>	<i>0.625 no</i>	<i>9.27E+00</i>	<i>195.3</i>	<i>193.8</i>	<i>193.8</i>
<i>Air Gap</i>	<i>air</i>	<i>0.000 no</i>	<i>1.73E-02</i>	<i>193.8</i>	<i>193.8</i>	<i>193.8</i>
Inner Shell	ss304	0.500 no	9.26E+00	193.8	192.7	192.7
Gamma Shield	lead	3.500 no	1.94E+01	192.7	189.1	189.1
Outer Shell	csteel	1.500 no	2.50E+01	189.1	188.0	188.0
Neutron Shield	ns3stif	3.000 no	9.49E-01	188.0	132.8	132.8
N. Shield Panel	ss304	0.125 no	8.90E+00	132.8	132.6	132.6

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Case 2C: 0° F, No solar, Horizontal w/ 0.75" air gap & 167" Cask Cavity Length  
 (Model to determine maximum DSC shell temp only)

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 0 °F  
 Solar Heat Load 0 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surface 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 167 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr·ft·°F)	k	Ti (°F)	To (°F)
DSC	ss304	0.250	no	1.03E+01	387.0	386.4
Air Gap	air	0.750	yes	1.95E-02	386.4	208.6
<i>Inner Shell</i>	<i>ss304</i>	<i>0.500</i>	<i>no</i>	<i>9.35E+00</i>	<i>208.6</i>	<i>207.3</i>
<i>Gamma Shield</i>	<i>lead</i>	<i>3.500</i>	<i>no</i>	<i>1.93E+01</i>	<i>207.3</i>	<i>203.4</i>
<i>Outer Shell</i>	<i>csteel</i>	<i>1.500</i>	<i>no</i>	<i>2.50E+01</i>	<i>203.4</i>	<i>202.2</i>
<i>Neutron Shield</i>	<i>ns3stif</i>	<i>3.000</i>	<i>no</i>	<i>9.49E-01</i>	<i>202.2</i>	<i>141.9</i>
<i>N. Shield Panel</i>	<i>ss304</i>	<i>0.125</i>	<i>no</i>	<i>8.95E+00</i>	<i>141.9</i>	<i>141.6</i>

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# Pacific Nuclear Fuel Services, Inc.

San Jose, California

PROJECT NUHOMS 10CFR72 CERTIFICATION FILE NO. NUH004.0407  
 CLIENT PNFSI

Case 3A: 70° F, Solar Load, Horizontal w/ 0.375" air gap  
 (Cask top model)

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 70 °F  
 Solar Heat Load 62 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surface 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 187.75 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr·ft·°F)	k	Ti (°F)	To (°F)
DSC	<i>ss304</i>	<i>0.625</i>	<i>no</i>	<i>1.04E+01</i>	<i>395.2</i>	<i>393.9</i>
Air Gap	<i>air</i>	<i>0.375</i>	<i>yes</i>	<i>2.02E-02</i>	<i>393.9</i>	<i>273.3</i>
Inner Shell	<i>ss304</i>	<i>0.500</i>	<i>no</i>	<i>9.69E+00</i>	<i>273.3</i>	<i>272.2</i>
Gamma Shield	<i>lead</i>	<i>3.500</i>	<i>no</i>	<i>1.89E+01</i>	<i>272.2</i>	<i>268.6</i>
Outer Shell	<i>csteel</i>	<i>1.500</i>	<i>no</i>	<i>2.47E+01</i>	<i>268.6</i>	<i>267.5</i>
Neutron Shield	<i>ns3stif</i>	<i>3.000</i>	<i>no</i>	<i>9.49E-01</i>	<i>267.5</i>	<i>212.8</i>
N. Shield Panel	<i>ss304</i>	<i>0.125</i>	<i>no</i>	<i>9.38E+00</i>	<i>212.8</i>	<i>212.6</i>

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# Pacific Nuclear Fuel Services, Inc.

San Jose, California

PROJECT NUHOMS 10CFR72 CERTIFICATION FILE NO. NUH004.0407  
 CLIENT PNFSI

Case 3B: 70° F, No Solar Load, Horizontal w/ no air gap  
 (Cask bottom model)

Number of Assemblies	24
Decay Heat per Assembly	1 kW
Ambient Temperature	70 °F
Solar Heat Load	0 Btu/hr·ft <sup>2</sup>
Emissivity of Outer Surfac	0.587
Cask Inner Radius	33 in
Cask Cavity Length	187.75 in
Orientation	Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr·ft·°F)	k	Ti (°F)	To (°F)
<i>DSC</i>	<i>ss304</i>	<i>0.625</i>	<i>no</i>	<i>9.60E+00</i>	<i>249.9</i>	<i>248.5</i>
<i>Air Gap</i>	<i>air</i>	<i>0.000</i>	<i>no</i>	<i>1.85E-02</i>	<i>248.5</i>	<i>248.5</i>
<i>Inner Shell</i>	<i>ss304</i>	<i>0.500</i>	<i>no</i>	<i>9.59E+00</i>	<i>248.5</i>	<i>247.5</i>
<i>Gamma Shield</i>	<i>lead</i>	<i>3.500</i>	<i>no</i>	<i>1.91E+01</i>	<i>247.5</i>	<i>243.8</i>
<i>Outer Shell</i>	<i>csteel</i>	<i>1.500</i>	<i>no</i>	<i>2.48E+01</i>	<i>243.8</i>	<i>242.7</i>
<i>Neutron Shield</i>	<i>ns3stif</i>	<i>3.000</i>	<i>no</i>	<i>9.49E-01</i>	<i>242.7</i>	<i>187.5</i>
<i>N. Shield Panel</i>	<i>ss304</i>	<i>0.125</i>	<i>no</i>	<i>9.22E+00</i>	<i>187.5</i>	<i>187.3</i>

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# Pacific Nuclear Fuel Services, Inc.

San Jose, California

PROJECT NUHOMS 10CFR72 CERTIFICATION FILE NO. NUH004.0407  
 CLIENT PNFSI

Case 3C: 70° F, Solar Load, Horizontal w/ 0.75" air gap & 167"  
 Cask Cavity Length  
 (Model to determine maximum DSC shell temp only)

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 70 °F  
 Solar Heat Load 62 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surface 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 167 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation	k (Btu/hr·ft·°F)	Ti (°F)	To (°F)
DSC	ss304	0.250	no	1.05E+01	433.6	433.1
Air Gap	air	0.750	yes	2.08E-02	433.1	287.8
Inner Shell	ss304	0.500	no	9.75E+00	287.8	286.6
Gamma Shield	lead	3.500	no	1.89E+01	286.6	282.6
Outer Shell	csteel	1.500	no	2.46E+01	282.6	281.4
Neutron Shield	ns3stif	3.000	no	9.49E-01	281.4	221.0
N. Shield Panel	ss304	0.125	no	9.43E+00	221.0	220.8

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San Jose, California

PROJECT NUHOMS 10CFR72 CERTIFICATION FILE NO. NUH004.0407  
 CLIENT PNFSI

Case 3D: 70° F, No Solar Load, Vertical w/ 0.375" air gap

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 70 °F  
 Solar Heat Load 0 Btu/hr-ft<sup>2</sup>  
 Emissivity of Outer Surfac 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 187.75 in  
 Orientation Vertical

Cask Layer	Material	Thickness (IN)	Radiation	k (Btu/hr-ft <sup>2</sup> ·°F)	Ti (°F)	To (°F)
DSC	ss304	0.625	no	1.02E+01	375.4	374.1
Air Gap	air	0.375	yes	1.97E-02	374.1	244.7
Inner Shell	ss304	0.500	no	9.57E+00	244.7	243.7
Gamma Shield	lead	3.500	no	1.91E+01	243.7	240.1
Outer Shell	csteel	1.500	no	2.48E+01	240.1	239.0
Neutron Shield	ns3stif	3.000	no	9.49E-01	239.0	184.3
N. Shield Panel	ss304	0.125	no	9.21E+00	184.3	184.1

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# Pacific Nuclear Fuel Services, Inc.

San Jose, California

PROJECT NUHOMS 10CFR72 CERTIFICATION FILE NO. NUH004.0407  
 CLIENT PNFSI

Case 4A: 100° F, Solar Load, Horizontal w/ 0.375" air gap  
 (Cask top model)

Number of Assemblies	24
Decay Heat per Assembly	1 kW
Ambient Temperature	100 °F
Solar Heat Load	62 Btu/hr·ft <sup>2</sup>
Emissivity of Outer Surfac	0.587
Cask Inner Radius	33 in
Cask Cavity Length	187.75 in
Orientation	Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr·ft·°F)	k	Ti (°F)	To (°F)
<i>DSC</i>	<i>ss304</i>	<i>0.625</i>	<i>no</i>	<i>1.04E+01</i>	<i>411.1</i>	<i>409.9</i>
<i>Air Gap</i>	<i>air</i>	<i>0.375</i>	<i>yes</i>	<i>2.06E-02</i>	<i>409.9</i>	<i>295.7</i>
Inner Shell	ss304	0.500	no	9.78E+00	295.7	294.7
Gamma Shield	lead	3.500	no	1.88E+01	294.7	291.0
Outer Shell	csteel	1.500	no	2.46E+01	291.0	289.9
Neutron Shield	ns3stif	3.000	no	9.49E-01	289.9	235.2
N. Shield Panel	ss304	0.125	no	9.51E+00	235.2	235.0

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 CLIENT PNFSI

Case 4B: 100° F, No Solar Load, Horizontal w/ no air gap

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 100 °F  
 Solar Heat Load 0 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surfac 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 187.75 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr·ft·°F)	k	Ti (°F)	To (°F)
DSC	ss304	0.625 no	<i>9.69E+00</i>	<i>273.4</i>	<i>272.1</i>	<i>272.1</i>
Air Gap	air	0.000 no	<i>1.89E-02</i>	<i>272.1</i>	<i>272.1</i>	<i>272.1</i>
Inner Shell	ss304	0.500 no	<i>9.69E+00</i>	<i>272.1</i>	<i>271.0</i>	<i>271.0</i>
Gamma Shield	lead	3.500 no	<i>1.90E+01</i>	<i>271.0</i>	<i>267.3</i>	<i>267.3</i>
Outer Shell	csteel	1.500 no	<i>2.47E+01</i>	<i>267.3</i>	<i>266.2</i>	<i>266.2</i>
Neutron Shield	ns3stif	3.000 no	<i>9.49E-01</i>	<i>266.2</i>	<i>211.0</i>	<i>211.0</i>
N. Shield Panel	ss304	0.125 no	<i>9.37E+00</i>	<i>211.0</i>	<i>210.8</i>	<i>210.8</i>

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San Jose, California

PROJECT NUHOMS 10CFR72 CERTIFICATION FILE NO. NUH004.0407  
 CLIENT PNFSI

Case 4C: 100° F, Solar Load, Horizontal w/ 0.75" air gap and 167" Cask Cavity Length  
 (Model to determine DSC maximum shell temp only)

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 100 °F  
 Solar Heat Load 62 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surfac 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 167 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr·ft·°F)	k	Ti (°F)	To (°F)
DSC	ss304	0.250	no	1.06E+01	447.7	447.2
Air Gap	air	0.750	yes	2.11E-02	447.2	309.9
Inner Shell	ss304	0.500	no	9.86E+00	309.9	308.8
Gamma Shield	lead	3.500	no	1.87E+01	308.8	304.7
Outer Shell	csteel	1.500	no	2.45E+01	304.7	303.5
Neutron Shield	ns3stif	3.000	no	9.49E-01	303.5	243.1
N. Shield Panel	ss304	0.125	no	9.56E+00	243.1	242.9

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 CLIENT PNFSI

Case 4D: 100° F, No Solar Load, Vertical w/ 0.375" air gap and 167" Cask Cavity Length

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 100 °F  
 Solar Heat Load 0 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surfac 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 167 in  
 Orientation Vertical

Cask Layer	Material	Thickness (IN)	Radiation	k (Btu/hr·ft·°F)	Ti (°F)	To (°F)
DSC	ss304	0.625	no	1.04E+01	412.6	411.2
Air Gap	air	0.375	yes	2.05E-02	411.2	283.3
Inner Shell	ss304	0.500	no	9.73E+00	283.3	282.1
Gamma Shield	lead	3.500	no	1.89E+01	282.1	278.1
Outer Shell	csteel	1.500	no	2.46E+01	278.1	276.9
Neutron Shield	ns3stif	3.000	no	9.49E-01	276.9	216.6
N. Shield Panel	ss304	0.125	no	9.40E+00	216.6	216.3

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 CLIENT PNFSI

Case 5A: 125° F, Max Solar Load, Horizontal w/ 0.375" air gap  
 (Cask top model)

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 125 °F  
 Solar Heat Load 123 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surfac 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 187.75 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr·ft·°F)	k	Ti (°F)	To (°F)
<i>DSC</i>	<i>ss304</i>	<i>0.625</i>	<i>no</i>	<i>1.06E+01</i>	<i>441.0</i>	<i>439.7</i>
<i>Air Gap</i>	<i>air</i>	<i>0.375</i>	<i>yes</i>	<i>2.13E-02</i>	<i>439.7</i>	<i>336.4</i>
<i>Inner Shell</i>	<i>ss304</i>	<i>0.500</i>	<i>no</i>	<i>1.00E+01</i>	<i>336.4</i>	<i>335.4</i>
<i>Gamma Shield</i>	<i>lead</i>	<i>3.500</i>	<i>no</i>	<i>1.86E+01</i>	<i>335.4</i>	<i>331.7</i>
<i>Outer Shell</i>	<i>csteel</i>	<i>1.500</i>	<i>no</i>	<i>2.43E+01</i>	<i>331.7</i>	<i>330.6</i>
<i>Neutron Shield</i>	<i>ns3stif</i>	<i>3.000</i>	<i>no</i>	<i>9.49E-01</i>	<i>330.6</i>	<i>275.9</i>
<i>N. Shield Panel</i>	<i>ss304</i>	<i>0.125</i>	<i>no</i>	<i>9.70E+00</i>	<i>275.9</i>	<i>275.7</i>

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 CLIENT PNFSI

Case 5B: 125° F, No Solar Load, Horizontal w/ no air gap  
 (Cask bottom model)

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 125 °F  
 Solar Heat Load 0 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surfac 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 187.75 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr·ft·°F)	k	Ti (°F)	To (°F)
DSC	ss304	0.625	<i>no</i>	<i>9.77E+00</i>	<i>293.1</i>	<i>291.7</i>
Air Gap	air	0.000	<i>no</i>	<i>1.93E-02</i>	<i>291.7</i>	<i>291.7</i>
Inner Shell	ss304	0.500	<i>no</i>	<i>9.76E+00</i>	<i>291.7</i>	<i>290.7</i>
Gamma Shield	lead	3.500	<i>no</i>	<i>1.88E+01</i>	<i>290.7</i>	<i>287.0</i>
Outer Shell	csteel	1.500	<i>no</i>	<i>2.46E+01</i>	<i>287.0</i>	<i>285.9</i>
Neutron Shield	ns3stif	3.000	<i>no</i>	<i>9.49E-01</i>	<i>285.9</i>	<i>230.7</i>
N. Shield Panel	ss304	0.125	<i>no</i>	<i>9.48E+00</i>	<i>230.7</i>	<i>230.4</i>

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 CLIENT PNFSI

Case 5C: 125° F, Solar Load, Horizontal w/ 0.75" air gap and 167" Cask Cavity Length  
 (Model to determine maximum DSC shell temp only)

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 125 °F  
 Solar Heat Load 123 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surface 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 167 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation (Btu/hr·ft·°F)	k	Ti (°F)	To (°F)
DSC	ss304	0.250	no	1.07E+01	474.4	473.8
Air Gap	air	0.750	yes	2.18E-02	473.8	349.8
Inner Shell	ss304	0.500	no	1.01E+01	349.8	348.7
Gamma Shield	lead	3.500	no	1.85E+01	348.7	344.6
Outer Shell	csteel	1.500	no	2.43E+01	344.6	343.3
Neutron Shield	ns3stif	3.000	no	9.49E-01	343.3	283.0
N. Shield Panel	ss304	0.125	no	9.73E+00	283.0	282.7

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 CLIENT PNFSI

Case 5D: 125° F, Max Solar Load, Horizontal w/ 0.375" air gap  
 and with no neutron shield  
 (Accident case)

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 125 °F  
 Solar Heat Load 123 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surface 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 187.75 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation	k (Btu/hr·ft·°F)	Ti (°F)	To (°F)
DSC	ss304	0.625	no	1.08E+01	486.7	485.5
Air Gap	air	0.375	yes	2.23E-02	485.5	396.1
Inner Shell	ss304	0.500	no	1.04E+01	396.1	395.1
Gamma Shield	lead	3.500	no	1.82E+01	395.1	391.3
Outer Shell	csteel	1.500	no	2.40E+01	391.3	390.2
Neutron Shield	airconv	3.000	yes	1.00E-01	390.2	275.9
N. Shield Panel	ss304	0.125	no	9.70E+00	275.9	275.7

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 CLIENT PNFSI

Case 5E: 125° F, Max Solar Load, Horizontal w/ 0.75" air gap  
 and with no neutron shield  
 (Accident case)

Number of Assemblies 24  
 Decay Heat per Assembly 1 kW  
 Ambient Temperature 125 °F  
 Solar Heat Load 123 Btu/hr·ft<sup>2</sup>  
 Emissivity of Outer Surface 0.587  
 Cask Inner Radius 33 in  
 Cask Cavity Length 167 in  
 Orientation Horizontal

Cask Layer	Material	Thickness (IN)	Radiation	k (Btu/hr·ft·°F)	Ti (°F)	To (°F)
DSC	ss304	0.250	no	1.10E+01	518.2	517.7
Air Gap	air	0.750	yes	2.28E-02	517.7	411.6
Inner Shell	ss304	0.500	no	1.04E+01	411.6	410.5
Gamma Shield	lead	3.500	no	1.81E+01	410.5	406.3
Outer Shell	csteel	1.500	no	2.39E+01	406.3	405.0
Neutron Shield	airconv	3.000	yes	1.00E-01	405.0	283.0
N. Shield Panel	ss304	0.125	no	9.73E+00	283.0	282.7

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 CLIENT PNFSI

### 3.7 Results for Cask/DSC Axial Cases

Case 1A: -40 F Ambient, Top End, No Solar, W/ Air Gap, 167" Cask Cavity.

(Model to determine the DSC axial temp distribution only)

Number of Assemblies	24	Internal Radiation	
Decay Heat per Assembly	1.00 kW	hr	1.175003
Ambient Temperature	-40 °F	hr	1.175003
Solar Heat Load	0 Btu/hr-ft <sup>2</sup>	ei	0.587
Emissivity of Outer Surface	0.587	eo	0.587
Cask Inner Radius	34.000 in		
Cask Cavity Length	187.750 in		
Convection Coefficient	1.033 Btu/hr-ft <sup>2</sup> -°F)		
Radiation Coefficient	0.516 Btu/hr-ft <sup>2</sup> -°F)		

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
DSC Plug	ss304	8.750	8.700	383.804	362.927
DSC Top Plate	ss304	1.250	8.700	362.927	359.944
Air Gap	air	0.500	0.020	359.944	208.534
Cask Struc Plate	csteel	3.000	25.000	208.534	206.043
Neutron Shield	NS3	2.000	0.488	206.043	121.003
Cask Top Plate	csteel	0.250	25.000	121.003	120.796

Note: Numbers and text in italic type are calculated for information only.

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# Pacific Nuclear Fuel Services, Inc.

San Jose, California

PROJECT NUHOMS 10CFR72 CERTIFICATION FILE NO. NUH004.0407  
 CLIENT PNFSI

Case 1B: -40 F Ambient, Top End, No Solar, No Air Gap, Cask layers only

Number of Assemblies	24	
Decay Heat per Assembly	1.00 kW	hr
Ambient Temperature	-40 °F	
Solar Heat Load	0	Btu/hr-ft <sup>2</sup>
Emissivity of Outer Surface	0.587	
Cask Inner Radius	34.000	in
Cask Cavity Length	187.750	in
Convection Coefficient	1.033	Btu/hr-ft <sup>2</sup> -°F)
Radiation Coefficient	0.516	Btu/hr-ft <sup>2</sup> -°F)

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
Top Cover Plate	csteel	3.000	25.000	208.534	206.043
Neutron Shield	ns3	2.000	0.488	206.043	121.003
Outer Plate	csteel	0.250	25.000	121.003	120.796

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# Pacific Nuclear Fuel Services, Inc.

San Jose, California

PROJECT NUHOMS 10CFR72 CERTIFICATION FILE NO. NUH004.0407  
 CLIENT PNFSI

Case 2A: 0 F Ambient, Top End, No Solar, W/ Air Gap, 167" Cask Cavity  
 (Model to determine the DSC axial temp distribution only)

Number of Assemblies	24	Internal Radiation	
Decay Heat per Assembly	1.00 kW	hr	1.295601
Ambient Temperature	0 °F	hr	1.295601
Solar Heat Load	0 Btu/hr-ft <sup>2</sup>	ei	0.587
Emissivity of Outer Surface	0.587	eo	0.587
Cask Inner Radius	34.000 in		
Cask Cavity Length	187.750 in		
Convection Coefficient	1.013 Btu/hr-ft <sup>2</sup> -°F)		
Radiation Coefficient	0.630 Btu/hr-ft <sup>2</sup> -°F)		

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
DSC Plug	ss304	8.750	8.703	403.271	382.402
DSC Top Plate	ss304	1.250	8.700	382.402	379.419
Air Gap	air	0.500	0.020	379.419	239.360
Cask Struc Plate	csteel	3.000	24.855	239.360	236.855
Neutron Shield	NS3	2.000	0.488	236.855	151.815
Cask Top Plate	csteel	0.250	25.000	151.815	151.608

Note: Numbers and text in italic type are included for information only.

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# Pacific Nuclear Fuel Services, Inc.

San Jose, California

PROJECT NUHOMS 10CFR72 CERTIFICATION FILE NO. NUH004.0407  
 CLIENT PNFSI

Case 2B: 0 F Ambient, Top End, No Solar, No Air Gap, Cask layers only.

Number of Assemblies	24	
Decay Heat per Assembly	1.00 kW	hr
Ambient Temperature	0 °F	
Solar Heat Load	0 Btu/hr-ft <sup>2</sup>	
Emissivity of Outer Surface	0.587	
Cask Inner Radius	34.000 in	
Cask Cavity Length	187.750 in	
Convection Coefficient	1.013 Btu/hr-ft <sup>2</sup> -°F)	
Radiation Coefficient	0.630 Btu/hr-ft <sup>2</sup> -°F)	

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
Top Cover Plate	csteel	3.000	24.855	239.360	236.855
Neutron Shield	ns3	2.000	0.488	236.855	151.815
Outer Plate	csteel	0.250	25.000	151.815	151.608

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# Pacific Nuclear Fuel Services, Inc.

San Jose, California

PROJECT NUHOMS 10CFR72 CERTIFICATION FILE NO. NUH004.0407  
 CLIENT PNFSI

Case 3A: 70 F Ambient, Top End, W/ Solar, W/ Air Gap, 167" Cask Cavity.

(Model to determine DSC axial temp distribution only)

Number of Assemblies	24	Internal Radiation	
Decay Heat per Assembly	1.00 kW	hr	1.678937
Ambient Temperature	70 °F	hr	1.678937
Solar Heat Load	62 Btu/hr-ft <sup>2</sup>	ei	0.587
Emissivity of Outer Surface	0.587	eo	0.587
Cask Inner Radius	34.000 in		
Cask Cavity Length	167.000 in		
Convection Coefficient	1.010 Btu/hr-ft <sup>2</sup> -°F)		
Radiation Coefficient	0.903 Btu/hr-ft <sup>2</sup> -°F)		

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
DSC Plug	ss304	8.750	8.951	467.704	445.316
DSC Top Plate	ss304	1.250	8.901	445.316	442.100
Air Gap	air	0.500	0.022	442.100	316.994
Cask Struc Plate	csteel	3.000	24.425	316.994	314.181
Neutron Shield	NS3	2.000	0.488	314.181	220.363
Cask Top Plate	csteel	0.250	24.954	220.363	220.134

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# Pacific Nuclear Fuel Services, Inc.

San Jose, California

PROJECT NUHOMS 10CFR72 CERTIFICATION FILE NO. NUH004.0407  
 CLIENT PNFSI

Case 3B: 70 F Ambient, Top End, W/ Solar, No Air Gap, Cask layers only.

Number of Assemblies	24	
Decay Heat per Assembly	1.00 kW	hr
Ambient Temperature	70 °F	
Solar Heat Load	62 Btu/hr-ft <sup>2</sup>	
Emissivity of Outer Surface	0.587	
Cask Inner Radius	34.000 in	
Cask Cavity Length	187.750 in	
Convection Coefficient	0.987 Btu/hr-ft <sup>2</sup> -°F)	
Radiation Coefficient	0.879 Btu/hr-ft <sup>2</sup> -°F)	

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
Top Cover Plate	csteel	3.000	24.530	297.902	295.364
Neutron Shield	ns3	2.000	0.488	295.364	210.324
Outer Plate	csteel	0.250	25.000	210.324	210.117

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# Pacific Nuclear Fuel Services, Inc.

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 CLIENT PNFSI

Case 4A: 100 F Ambient, Top End, W/ Solar, W/ Air Gap, 167" Cask Cavity.

(Model to determine DSC axial shell temps only)

Number of Assemblies	24	Internal Radiation	
Decay Heat per Assembly	1.00 kW	hr	1.794342
Ambient Temperature	100 °F	hr	1.794342
Solar Heat Load	62 Btu/hr-ft <sup>2</sup>	ei	0.587
Emissivity of Outer Surface	0.587	eo	0.587
Cask Inner Radius	34.000 in		
Cask Cavity Length	167.000 in		
Convection Coefficient	0.992 Btu/hr-ft <sup>2</sup> -°F)		
Radiation Coefficient	1.023 Btu/hr-ft <sup>2</sup> -°F)		

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
DSC Plug	ss304	8.750	9.011	483.206	460.969
DSC Top Plate	ss304	1.250	8.962	460.969	457.775
Air Gap	air	0.500	0.022	457.775	339.382
Cask Struc Plate	csteel	3.000	24.300	339.382	336.554
Neutron Shield	NS3	2.000	0.488	336.554	242.736
Cask Top Plate	csteel	0.250	24.830	242.736	242.505

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PROJECT NUHOMS 10CFR72 CERTIFICATION FILE NO. NUH004.0407  
 CLIENT PNFSI

Case 4B: 100 F Ambient, Top End, W/ Solar, No Air Gap, Cask layers only.

Number of Assemblies	24		
Decay Heat per Assembly	1.00 kW	hr	ERR
Ambient Temperature	100 °F		
Solar Heat Load	62 Btu/hr-ft <sup>2</sup>		
Emissivity of Outer Surface	0.587		
Cask Inner Radius	34.000 in		
Cask Cavity Length	187.750 in		
Convection Coefficient	0.970 Btu/hr-ft <sup>2</sup> -°F)		
Radiation Coefficient	0.998 Btu/hr-ft <sup>2</sup> -°F)		

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
Top Cover Plate	csteel	3.000	24.403	320.682	318.130
Neutron Shield	ns3	2.000	0.488	318.130	233.091
Outer Plate	csteel	0.250	24.883	233.091	232.882

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# Pacific Nuclear Fuel Services, Inc.

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 CLIENT PNFSI

Case 4C: 100 F Ambient, Top End, No Solar, w/ 0.5" Air Gap.

Number of Assemblies	24	Internal Radiation	
Decay Heat per Assembly	1.00 kW	hr	1.814545
Ambient Temperature	100 °F	hr	1.814545
Solar Heat Load	0 Btu/hr-ft <sup>2</sup>	ei	0.587
Emissivity of Outer Surface	0.587	eo	0.587
Cask Inner Radius	34.000 in		
Cask Cavity Length	167.000 in		
Convection Coefficient	1.116 Btu/hr-ft <sup>2</sup> -°F)		
Radiation Coefficient	0.992 Btu/hr-ft <sup>2</sup> -°F)		

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
DSC Plug	ss304	7.000	9.018	482.711	464.935
DSC Top Plate	ss304	1.750	8.975	464.935	460.470
Air Gap	air	0.500	0.022	460.470	343.176
Bottom Struc Plate	ss304	1.750	8.700	343.176	338.569
Neutron Shield	NS3	2.250	0.488	338.569	233.023
Bottom Plate	ss304	1.000	8.700	233.023	230.391

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# Pacific Nuclear Fuel Services, Inc.

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 CLIENT PNFSI

Case 5A: 125 F Ambient, Top End, W/ Solar, W/ Air Gap, 167" Cask Cavity.

(Model to determine DSC axial shell temps only)

Number of Assemblies	24	Internal Radiation	
Decay Heat per Assembly	1.00 kW	hr	1.919935
Ambient Temperature	125 °F	hr	1.919935
Solar Heat Load	123 Btu/hr-ft <sup>2</sup>	ei	0.587
Emissivity of Outer Surface	0.587	eo	0.587
Cask Inner Radius	34.000 in		
Cask Cavity Length	167.000 in		
Convection Coefficient	0.988 Btu/hr-ft <sup>2</sup> -°F)		
Radiation Coefficient	1.143 Btu/hr-ft <sup>2</sup> -°F)		

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
DSC Plug	ss304	8.750	9.075	499.549	477.469
DSC Top Plate	ss304	1.250	9.026	477.469	474.297
Air Gap	air	0.500	0.022	474.297	362.415
Cask Struc Plate	csteel	3.000	24.172	362.415	359.573
Neutron Shield	NS3	2.000	0.488	359.573	265.754
Cask Top Plate	csteel	0.250	24.702	265.754	265.523

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# Pacific Nuclear Fuel Services, Inc.

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 CLIENT PNFSI

Case 5B: 125 F Ambient, Top End, W/ Solar, No Air Gap, Cask layers only.

Number of Assemblies	24	
Decay Heat per Assembly	1.00 kW	hr
Ambient Temperature	125 °F	
Solar Heat Load	123 Btu/hr-ft <sup>2</sup>	
Emissivity of Outer Surface	0.587	
Cask Inner Radius	34.000 in	
Cask Cavity Length	187.750 in	
Convection Coefficient	0.966 Btu/hr-ft <sup>2</sup> -°F)	
Radiation Coefficient	1.118 Btu/hr-ft <sup>2</sup> -°F)	

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
Top Cover Plate	csteel	3.000	24.273	344.178	341.613
Neutron Shield	ns3	2.000	0.488	341.613	256.573
Outer Plate	csteel	0.250	24.753	256.573	256.364

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# Pacific Nuclear Fuel Services, Inc.

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 CLIENT PNFSI

Case 6A: -40 F Ambient, Bottom End, No Solar, W/ Air Gap, 167" Cask Cavity.

(Model to determine DSC axial temp distribution only)

Number of Assemblies	24	Internal Radiation	
Decay Heat per Assembly	1.00 kW	hr	1.351262
Ambient Temperature	-40 °F	hr	1.351262
Solar Heat Load	0 Btu/hr-ft <sup>2</sup>	ei	0.587
Emissivity of Outer Surface	0.587	eo	0.587
Cask Inner Radius	34.000 in		
Cask Cavity Length	167.000 in		
Convection Coefficient	1.058 Btu/hr-ft <sup>2</sup> -°F)		
Radiation Coefficient	0.536 Btu/hr-ft <sup>2</sup> -°F)		

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
DSC Plug	ss304	7.000	8.764	417.603	399.312
DSC Top Plate	ss304	1.750	8.719	399.312	394.716
Air Gap	air	0.500	0.020	394.716	245.326
Bottom Struc Plate	ss304	1.750	8.700	245.326	240.719
Neutron Shield	NS3	2.250	0.488	240.719	135.173
Bottom Plate	ss304	1.000	8.439	135.173	132.460

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 CLIENT PNFSI

Case 6B: -40 F Ambient, Bottom End, No Solar, No Air Gap, Cask Layers only.

Number of Assemblies	24	
Decay Heat per Assembly	1.00 kW	hr
Ambient Temperature	-40 °F	
Solar Heat Load	0	Btu/hr-ft <sup>2</sup>
Emissivity of Outer Surface	0.587	
Cask Inner Radius	34.000	in
Cask Cavity Length	187.750	in
Convection Coefficient	1.033	Btu/hr-ft <sup>2</sup> -°F)
Radiation Coefficient	0.516	Btu/hr-ft <sup>2</sup> -°F)

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
Top Cover Plate	csteel	1.750	24.967	218.750	217.295
Neutron Shield	ns3	2.250	0.488	217.295	121.626
Outer Plate	csteel	1.000	25.000	121.626	120.796

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# Pacific Nuclear Fuel Services, Inc.

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 CLIENT PNFSI

Case 7A: 0 F Ambient, Bottom End, No Solar, W/ Air Gap, 167" Cask Cavity.

(Model to determine DSC axial temp distribution only)

Number of Assemblies	24	Internal Radiation	
Decay Heat per Assembly	1.00 kW	hr	1.481999
Ambient Temperature	0 °F	hr	1.481999
Solar Heat Load	0 Btu/hr-ft <sup>2</sup>	ei	0.587
Emissivity of Outer Surface	0.587	eo	0.587
Cask Inner Radius	34.000 in		
Cask Cavity Length	167.000 in		
Convection Coefficient	1.037 Btu/hr-ft <sup>2</sup> -°F)		
Radiation Coefficient	0.651 Btu/hr-ft <sup>2</sup> -°F)		

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
DSC Plug	ss304	7.000	8.839	436.853	418.717
DSC Top Plate	ss304	1.750	8.795	418.717	414.160
Air Gap	air	0.500	0.021	414.160	275.568
Bottom Struc Plate	ss304	1.750	8.700	275.568	270.961
Neutron Shield	NS3	2.250	0.488	270.961	165.416
Bottom Plate	ss304	1.000	8.540	165.416	162.734

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# Pacific Nuclear Fuel Services, Inc.

San Jose, California

PROJECT NUHOMS 10CFR72 CERTIFICATION FILE NO. NUH004.0407  
 CLIENT PNFSI

Case 7B: 0 F Ambient, Bottom End, No Solar, No Air Gap, Cask layers only.

Number of Assemblies	24	
Decay Heat per Assembly	1.00 kW	hr
Ambient Temperature	0 °F	
Solar Heat Load	0	Btu/hr-ft <sup>2</sup>
Emissivity of Outer Surface	0.587	
Cask Inner Radius	34.000 in	
Cask Cavity Length	187.750 in	
Convection Coefficient	1.013	Btu/hr-ft <sup>2</sup> -°F)
Radiation Coefficient	0.630	Btu/hr-ft <sup>2</sup> -°F)

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
Top Cover Plate	csteel	1.750	24.795	249.572	248.107
Neutron Shield	ns3	2.250	0.488	248.107	152.438
Outer Plate	csteel	1.000	25.000	152.438	151.608

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 CLIENT PNFSI

Case 8A: 70 F Ambient, Bottom End, W/ Solar, W/ Air Gap, 167" Cask Cavity.

(Model to determine DSC axial temp distribution only)

Number of Assemblies	24	Internal Radiation	
Decay Heat per Assembly	1.00 kW	hr	1.760358
Ambient Temperature	70 °F	hr	1.760358
Solar Heat Load	62 Btu/hr-ft <sup>2</sup>	ei	0.587
Emissivity of Outer Surface	0.587	eo	0.587
Cask Inner Radius	34.000 in		
Cask Cavity Length	167.000 in		
Convection Coefficient	1.010 Btu/hr-ft <sup>2</sup> -°F)		
Radiation Coefficient	0.903 Btu/hr-ft <sup>2</sup> -°F)		

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
DSC Plug	ss304	7.000	8.990	475.521	457.690
DSC Top Plate	ss304	1.750	8.947	457.690	453.210
Air Gap	air	0.500	0.022	453.210	332.918
Bottom Struc Plate	ss304	1.750	8.700	332.918	328.312
Neutron Shield	NS3	2.250	0.488	328.312	222.766
Bottom Plate	ss304	1.000	8.700	222.766	220.134

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 CLIENT PNFSI

Case 8B: 70 F Ambient, Bottom End, W/ Solar, No Air Gap, Cask layers only.

Number of Assemblies	24	
Decay Heat per Assembly	1.00 kW	hr
Ambient Temperature	70 °F	
Solar Heat Load	62 Btu/hr-ft <sup>2</sup>	
Emissivity of Outer Surface	0.587	
Cask Inner Radius	34.000 in	
Cask Cavity Length	187.750 in	
Convection Coefficient	0.987 Btu/hr-ft <sup>2</sup> -°F)	
Radiation Coefficient	0.879 Btu/hr-ft <sup>2</sup> -°F)	

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
Top Cover Plate	csteel	1.750	24.470	308.101	306.616
Neutron Shield	ns3	2.250	0.488	306.616	210.947
Outer Plate	csteel	1.000	25.000	210.947	210.117

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# Pacific Nuclear Fuel Services, Inc.

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 CLIENT PNFSI

Case 9A: 100 F Ambient, Bottom End, W/ Solar, W/ Air Gap, 167" Cask Cavity.

(Model to determine DSC axial temp distribution only)

Number of Assemblies	24	Internal Radiation	
Decay Heat per Assembly	1.00 kW	hr	1.880328
Ambient Temperature	100 °F	hr	1.880328
Solar Heat Load	62 Btu/hr-ft <sup>2</sup>	ei	0.587
Emissivity of Outer Surface	0.587	eo	0.587
Cask Inner Radius	34.000 in		
Cask Cavity Length	167.000 in		
Convection Coefficient	0.992 Btu/hr-ft <sup>2</sup> -°F)		
Radiation Coefficient	1.023 Btu/hr-ft <sup>2</sup> -°F)		

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
DSC Plug	ss304	7.000	9.052	491.304	473.594
DSC Top Plate	ss304	1.750	9.009	473.594	469.145
Air Gap	air	0.500	0.022	469.145	355.290
Bottom Struc Plate	ss304	1.750	8.700	355.290	350.683
Neutron Shield	NS3	2.250	0.488	350.683	245.138
Bottom Plate	ss304	1.000	8.700	245.138	242.505

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San Jose, California

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Case 9B: 100 F Ambient, Bottom End, W/ Solar, No Air Gap, Cask layers only.

Number of Assemblies	24	
Decay Heat per Assembly	1.00 kW	hr
Ambient Temperature	100 °F	
Solar Heat Load	62 Btu/hr-ft <sup>2</sup>	
Emissivity of Outer Surface	0.587	
Cask Inner Radius	34.000 in	
Cask Cavity Length	187.750 in	
Convection Coefficient	0.970 Btu/hr-ft <sup>2</sup> -°F)	
Radiation Coefficient	0.998 Btu/hr-ft <sup>2</sup> -°F)	

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
Top Cover Plate	csteel	1.750	24.344	330.878	329.386
Neutron Shield	ns3	2.250	0.488	329.386	233.717
Outer Plate	csteel	1.000	24.882	233.717	232.882

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Case 9B: 100 F Ambient, Bottom End, No Solar, w/ 0.5" Air Gap.

Number of Assemblies	24	Internal Radiation	
Decay Heat per Assembly	1.00 kW	hr	1.222544
Ambient Temperature	100 °F	hr	1.222544
Solar Heat Load	0 Btu/hr-ft <sup>2</sup>	ei	0.587
Emissivity of Outer Surface	0.587	eo	0.587
Cask Inner Radius	34.000 in		
Cask Cavity Length	167.000 in		
Convection Coefficient	0.000 Btu/hr-ft <sup>2</sup> -°F)		
Radiation Coefficient	0.705 Btu/hr-ft <sup>2</sup> -°F)		

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
DSC Plug	ss304	7.000	8.700	397.821	379.395
DSC Top Plate	ss304	1.750	8.700	379.395	374.788
Air Gap	air	0.500	0.020	374.788	212.904
Bottom Struc Plate	ss304	1.750	8.695	212.904	208.295
Neutron Shield	NS3	2.250	0.488	208.295	102.749
Bottom Plate	ss304	1.000	8.331	102.749	100.000

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Case 10A: 125 F Ambient, Bottom End, W/ Solar, W/ Air Gap, 167" Cask Cavity.

(Model to determine DSC axial temp distribution only)

Number of Assemblies	24	Internal Radiation	
Decay Heat per Assembly	1.00 kW	hr	2.010767
Ambient Temperature	125 °F	hr	2.010767
Solar Heat Load	123 Btu/hr-ft <sup>2</sup>	ei	0.587
Emissivity of Outer Surface	0.587	eo	0.587
Cask Inner Radius	34.000 in		
Cask Cavity Length	167.000 in		
Convection Coefficient	0.988 Btu/hr-ft <sup>2</sup> -°F)		
Radiation Coefficient	1.143 Btu/hr-ft <sup>2</sup> -°F)		

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
DSC Plug	ss304	7.000	9.117	507.921	490.337
DSC Top Plate	ss304	1.750	9.074	490.337	485.921
Air Gap	air	0.500	0.023	485.921	378.307
<i>Bottom Struc Plate</i>	<i>ss304</i>	<i>1.750</i>	<i>8.700</i>	<i>378.307</i>	<i>373.701</i>
<i>Neutron Shield</i>	<i>NS3</i>	<i>2.250</i>	<i>0.488</i>	<i>373.701</i>	<i>268.155</i>
<i>Bottom Plate</i>	<i>ss304</i>	<i>1.000</i>	<i>8.700</i>	<i>268.155</i>	<i>265.523</i>

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Case 10B: 125 F Ambient, Bottom End, W/ Solar, No Air Gap, Cask layers only.

Number of Assemblies	24	
Decay Heat per Assembly	1.00 kW	hr
Ambient Temperature	125 °F	
Solar Heat Load	123 Btu/hr-ft <sup>2</sup>	
Emissivity of Outer Surface	0.587	
Cask Inner Radius	34.000 in	
Cask Cavity Length	187.750 in	
Convection Coefficient	0.966 Btu/hr-ft <sup>2</sup> -°F)	
Radiation Coefficient	1.118 Btu/hr-ft <sup>2</sup> -°F)	

Cask Layer	Material	Thickness (in)	k (Btu/hr-ft-°F)	Ti (°F)	To (°F)
Top Cover Plate	csteel	1.750	24.213	354.372	352.872
Neutron Shield	ns3	2.250	0.488	352.872	257.202
Outer Plate	csteel	1.000	24.751	257.202	256.364

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## 4.0 REFERENCES

- 1 PNFSI Calculation Package titled "Duke NUHOMS<sup>®</sup>-24P HSM Heat Transfer Analysis for 8, 10 and 16 Year Old Fuel," Calculation No. DUK003-204, Revision 0, File No. DUK003.0204.
- 2 Holman, J. P., "Heat Transfer," Fourth-Edition, McGraw-Hill Book Company, 1976.
- 3 Lienhard, John H., "A Heat Transfer Textbook," Second Edition, Prentice-Hall, Inc., 1987.
- 4 Rohsenow, Warren M., and Harry Choi, "Heat, Mass, and Momentum Transfer," Prentice-Hall, Inc., 1961.
- 5 "HTAS1: A Two-dimensional Heat Transfer Analysis of Fuel Casks," NUREG/CR-0200, Volume 1, Section H1, ORNL/NUREG/CSD-2/V1/R3.
- 6 PNFSI Calculation Package titled, "Thermal Analysis of the DSC/Cask Geometry with Various Solid Neutron Shield Materials for NUHOMS<sup>®</sup>-24P Design," Calculation number DUK003.0215, Revision 0, File number DUK003.0215.
- 7 "IF-300 Shipping Cask Consolidated Safety Analysis Report," General Electric Company, NEDO-10084-3, February, 1985.
- 8 U. S. Government, "Packaging and Transportation of Radioactive Material," Title 10 Code of Federal Regulations, Part 71, Office of the Federal Register, Washington, D.C. (1988)
- 9 PNFSI Report Titled "Safety Analysis Report for the Standardized NUHOMS<sup>®</sup> Horizontal Modular Storage System For Irradiated Fuel," PNFSI Document number NUH-003, Revision 0, PNFSI File No. NUH003.0103.
- 10 Letter for J. K. Wooley (Bisco Products) to B. D. Thomas (NUTECH/PNFSI) dated January 8, 1987, PNFSI File No. NUH004.0012.
- 11 1989 ASHRAE Fundamentals Handbook, I-P Edition, American Society of Heating, Refrigerating and Air Conditioning Engineers, Inc.

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