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October 1, 1985

Director, Office of Nuclear Reactor Regulation

Attention: Mr. J. A. Zwolinski, Chief  
Operating Reactors Branch No. 5  
Division of Licensing

U. S. Nuclear Regulatory Commission  
Washington, D.C. 20555

Gentlemen:

Subject: Docket No. 50-206  
NUREG-0737 Item II.D.1-Performance Testing of  
Relief and Safety Valves  
San Onofre Nuclear Generating Station  
Unit 1

References: 1. Letter, D. M. Crutchfield, NRC, to K. Baskin, SCE  
NUREG-0737 Item II.D.1-Performance Testing of  
Relief and Safety Valves, December 19, 1983

2. Letter, R. W. Krieger, SCE to D. M. Crutchfield, NRC,  
NUREG-0737, Item II.D.1-Performance Testing of  
Relief and Safety Valves, February 22, 1984

Reference 1 indicated that you had reviewed our December 2, 1982 and July 13, 1983 submittals and requested that we provide additional information regarding the performance testing of relief and safety valves. We responded in Reference 2 by stating that we would require additional time to prepare our responses and that a schedule would be provided after return to service from the seismic backfit outage. Accordingly, the additional information requested by Reference 1 is provided as an Enclosure to this letter. The information provided in the Enclosure should resolve the subject TMI Action Plan item for San Onofre Unit 1.

If you have any questions, please let me know.

Very truly yours,

*M. O. Medford*

AO46  
1/1

Enclosures

8510100104 851001  
PDR ADOCK 05000206 PDR  
P

NOTE: COMPARISON  
M16B 2 C45  
POP 1 C4  
211R 1 C4  
USCL 1 C4  
OAB 1 C4

Enclosure

RESPONSES TO NRC QUESTIONS ON  
PRESSURIZER RELIEF AND SAFETY VALVE TESTING  
SAN ONOFRE NUCLEAR GENERATING STATION  
UNIT 1

The information provided below responds to the Request for Additional Information, TMI Action NUREG-0737 (II.D.1), Relief and Safety Valve Testing for San Onofre Unit 1, November 1983.

NRC Question No. 1

The submittal treats a steam flow discharge through the safety valves corresponding to a Locked Rotor event as the limiting overpressure transient. It does not discuss, though, whether single failures after the initiating event were considered that could lead to water flow through the valves. Provide such a discussion on single failures to show how the NUREG-0737 requirement that single failures be chosen so as to maximize dynamic loads on the safety/relief valves has been met.

SCE Response

The limiting overpressure transient that incurs safety valve actuation is the Loss of External Load event (Final Engineering Report and Safety Analysis (FERSA) 10.6). The Loss of External Load analysis assumes an initial core power of 103 percent of rated with no direct reactor trip on turbine trip. The pressurizer spray, power-operated relief valves and steam release to the atmosphere and condenser are assumed inoperable. The combined effect from these assumptions produces the greatest (fastest) reactor coolant system pressurization rate.

Since the peak pressure is observed within a few seconds of the transient initiation, single failures within the engineered safeguards systems would have little or no effect on the pressurization rate or observed peak pressure.

NRC Question No. 2

Overpressure transients cause the pressurizer sprays to activate which adds moisture to the steam volume. When the safety or relief valves open they would thus pass a steam-water mixture. Explain whether this effect was considered in selecting the transient that produces maximum loading on the system.

SCE Response

The operation of pressurizer spray will not increase the valve and discharge piping loads because the peak load occurs prior to the time when any wet steam due to entrained spray can reach the safety valve.

As mentioned in the response to Question 9, the maximum discharge piping loads occur upon valve opening. This means that the peak force in a given piping segment occurs when the initial pressure surge due to valve opening reaches that segment. The inlet piping for the safety valve and the pressurizer volume above the spray header will initially contain saturated steam. In order for any postulated wet steam to reach the discharge piping, the initial quantity of saturated steam must pass through the safety valve. For San Onofre Unit 1 this would take at least 0.0165 seconds after the valve initially opens. By the time the postulated wet steam reaches the valve, the valve is fully open and the initial pressure surge has already occurred. This is further substantiated by EPRI safety valve test data for steam to water transition tests. In these tests the safety valve actuated on saturated steam, followed by a transition of saturated water after the valve opened. The peak loads occurred when the valve initially opened prior to the transition to water. Therefore, the operation of pressurizer spray will not result in discharge piping loads in excess of those values previously calculated.

As stated in the response to Question 7, the bending moments predicted to act on the safety valve discharge flange in the San Onofre piping analysis are significantly less than those measured during the test program. Therefore, the operability of the safety valves is not impaired by the calculated piping loads.

NRC Question No. 3

Results from the EPRI steam tests on the Crosby 3K6 safety valves indicate that blowdowns may exceed the 4% value from the valve specification, depending on the ring settings used (see related Question 6). If expected plant blowdowns do exceed 4%, these higher blowdowns could cause a rise in pressurizer water level such that water may reach the safety valve inlet line and result in a steam-water flow situation. Additionally, the pressure might be sufficiently decreased that adequate cooling might not be achieved for decay heat removal. Discuss these consequences of higher blowdowns if increased blowdowns are expected.

SCE Response

The impact on plant safety of pressurizer relief valve blowdowns in excess of 4% for San Onofre Unit 1 has been evaluated. The results of this evaluation show no adverse effects on plant safety.

Relief valve blowdowns in excess of that assumed in the San Onofre Nuclear Generating Station Unit 1 (SONGS 1) Final Engineering Report and Safety Analysis (FERSA) will have the following effects on the events in which relief valve actuation occurs:

- a) Increased pressurizer water level during and following the valve blowdown;
- b) Lower pressurizer pressure during and following valve blowdown;
- c) Increased inventory loss through the relief valve.

The impact of the increased relief valve blowdowns with respect to the above effects was evaluated for the single SONGS 1 FERSA event in which relief valve actuation occurs (i.e., Loss of Load).

For the Loss of Load event, results from sensitivity analyses performed for 4 loop plants were used for the evaluation. It is felt that similar results would be found for 3 loop plants. These analyses investigated the effects of different blowdown rates on the Loss of Load event. The results showed only marginal increases in pressurizer water volume and the maximum pressurizer water levels were well below the level at which liquid relief would occur. Peak RCS pressures were shown to be unaffected by the increased blowdowns. The increased blowdowns did result in lower pressurizer pressure and increased RCS inventory loss; however, these had no adverse impact on the event and adequate decay heat removal was maintained.

NRC Question No. 4

In discussion on valve inlet fluid conditions for low temperature overpressurization transients, the submittal identifies expected fluid conditions for a water discharge transient. According to the Westinghouse report on valve inlet fluid conditions, however, the fluid conditions for cold overpressurization events vary between steam and water. To assure that the relief valves operate under all of these events, discuss the range of fluid conditions expected for the varying types of fluid discharge and identify the test data that demonstrate operability over the range of conditions. Verify that the fluid conditions were properly enveloped during the tests. Confirm that the high pressure steam tests demonstrate valve operability for the low pressure steam case for both opening and closing of the relief valve.

SCE Response

The maximum temperature and pressure conditions that can be achieved at the PORV inlet coincidentally occur for steam bubble operation. Since pressure is normally maintained below the PORV setpoint, the maximum steam and saturated liquid pressure maintained in the pressurizer during startup and shutdown operations in anticipation of the cold overpressurization (COP) event would occur at the PORV setpoint. This pressure ( $P'$ ) and corresponding temperature ( $T'$ ) would be as follows:

<u>Plant</u>	<u><math>P'</math> (psig)</u>	<u><math>T'</math> (deg F)</u>
San Onofre Unit 1	500	470

Using these conditions, the potential worst case scenarios for PORV discharge during a COP event would be:

1. Discharge of saturated steam at  $P \leq P'$  and  $T \leq T'$  (steam in upper phase of pressurizer)
2. Discharge of saturated water at  $P \leq P'$  and  $T \leq T'$  (saturated water in pressurizer)

3. Discharge of subcooled water at  $P < P'$  and  $T < T'$  (mixing of colder RCS water with saturated pressurizer water)
4. Scenario 1 followed by Scenario 2
5. Scenario 2 followed by Scenario 3
6. Scenario 1 followed by Scenario 2 followed by Scenario 3.

EPRI test conditions for PORV's were chosen based on expected fluid conditions. Tests were limited but designed to confirm operability over a full range of expected inlet conditions. Steam, steam to water and water flow tests were conducted. Results of these tests can be found in EPRI report EPRI NP-2670-LD, Volume 9, Table IX-3. Although steam tests were conducted only at high pressures, it is expected that satisfactory performance would also result at the less severe lower pressures. This can be confirmed by the high pressure versus low pressure water tests where successful valve operation was observed.

NRC Question No. 5

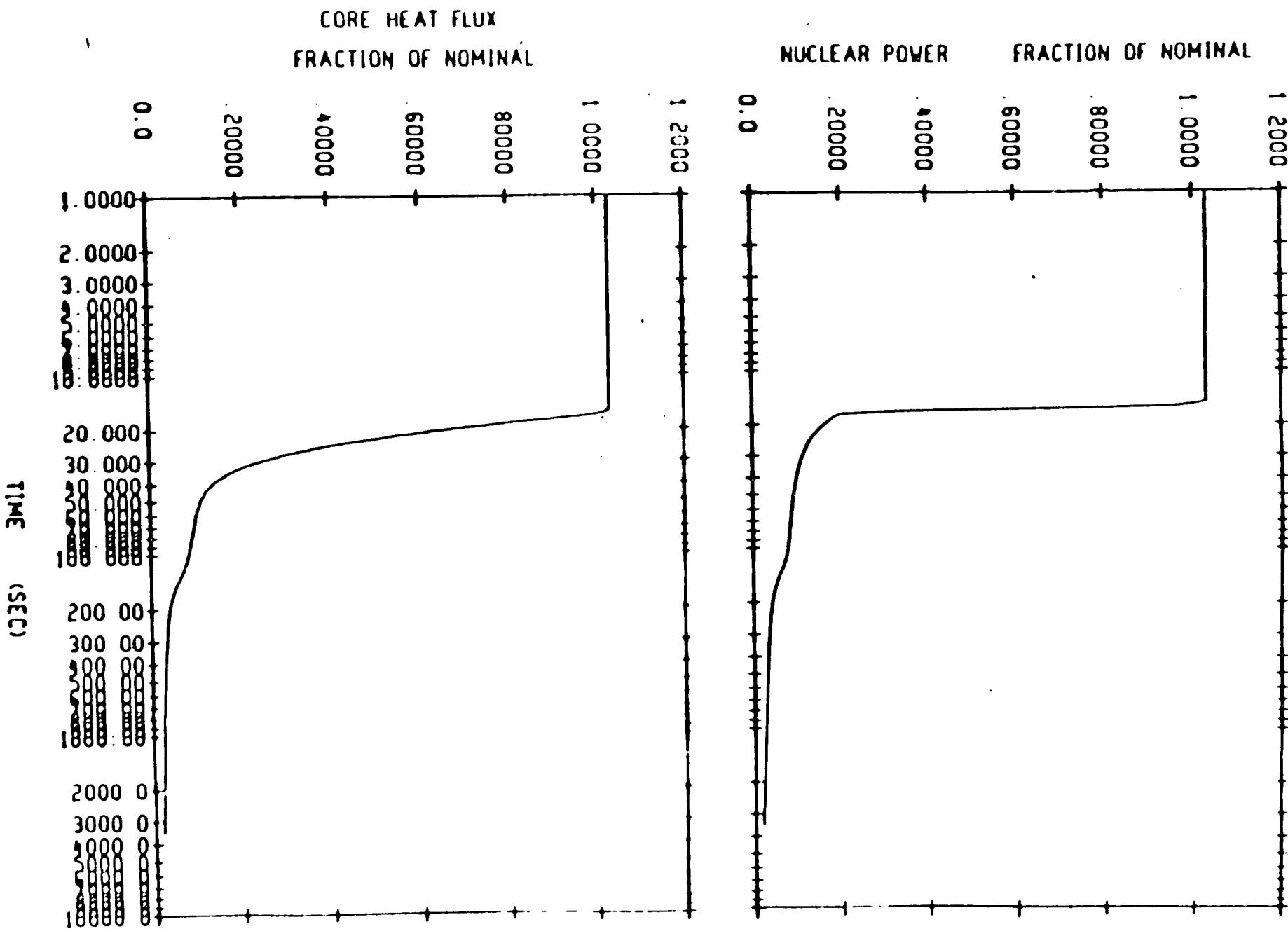
The Westinghouse valve inlet fluid conditions report stated that liquid discharge through both the safety and relief valves is predicted for an FSAR feedline break event. The Westinghouse report gave expected peak pressures and pressurization rates for some plants having an FSAR feedline break analysis. San Onofre 1 was not included in this list of plants having such an FSAR analysis. Nor does the submittal address the feedline break event. NUREG-0737, however, requires analysis of accidents and occurrences referenced in Regulatory Guide 1.70, Revision 2, and one of the accidents so required is the feedline break. Provide fluid pressure and pressurization rate, fluid temperature, valve flow rate, and the time duration for the feedline break event. Provide assurance that the valves passed water for a sufficient duration during the tests to cover this event and furnish an analysis which demonstrates safety of the safety/relief valve system for the feedline break event.

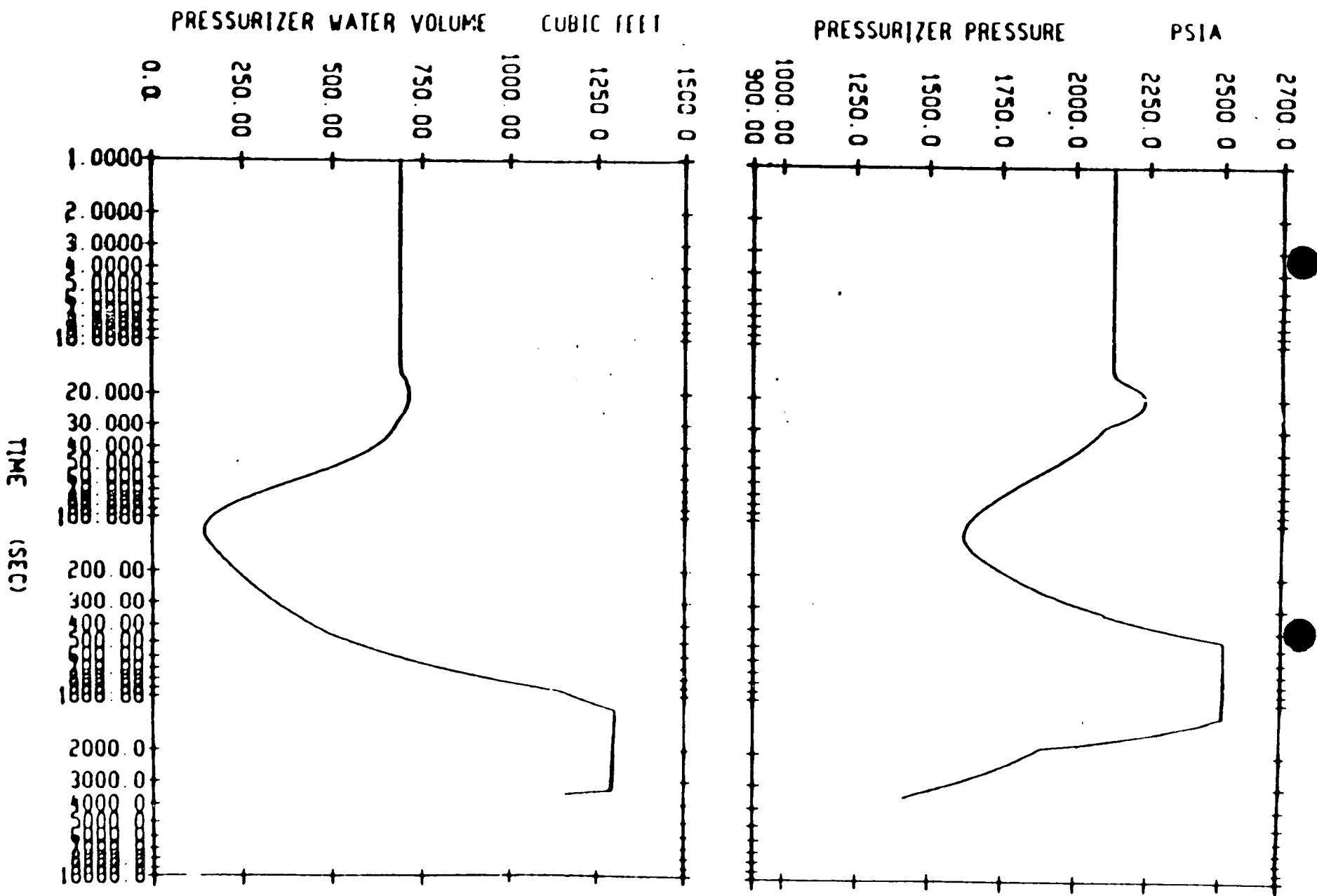
SCE Response

In response to post-TMI requirements, a feedline break analysis was performed and submitted to the NRC under the subject of Automatic Initiation of Auxiliary Feedwater Systems by letters dated March 6, 1981 and November 18, 1981. This analysis was reviewed and approved by the NRC as part of TMI Item No. II.E.1.1 and documented in NRC Amendment No. 65 dated October 2, 1982. The feedline break analysis was also reviewed and approved by the NRC as part of SEP Topic XV-6, Feedwater System Pipe Breaks Inside and Outside Containment (PWR), letter dated March 3, 1982 and XV-5, Loss of Normal Feedwater Flow, letter dated March 14, 1983.

The limiting feedline break analysis for San Onofre Unit 1 assumes auxiliary feedwater flowrate of 250 gpm and operator action time of 15 minutes after reactor trip. The appropriate plots of pressure, temperature and flowrates are attached. The defined "acceptable results" considered no bulk boiling in

the primary system prior to the time of loop temperature turnaround, i.e., the time when the heat removal rate equals the heat input on the primary side. Note that the peak pressurizer water volume is approximately 1299 cubic feet and the pressurizer capacity is 1300 cubic feet. Based upon these results, the limiting feedline break transient does not result in a solid pressurizer; thus, water relief is not predicted.

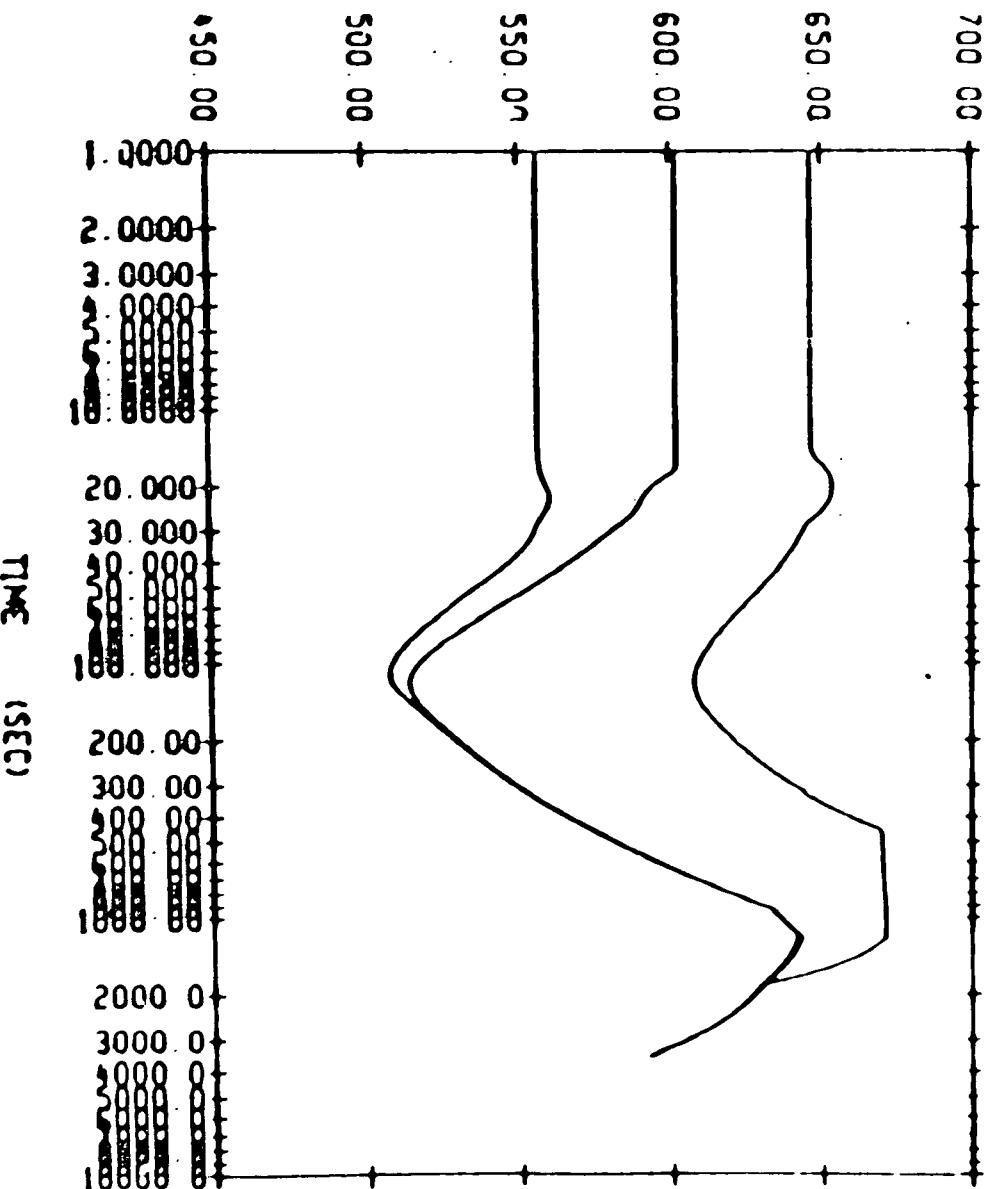




Feedline Rupture 250 gpm @ 15 min.

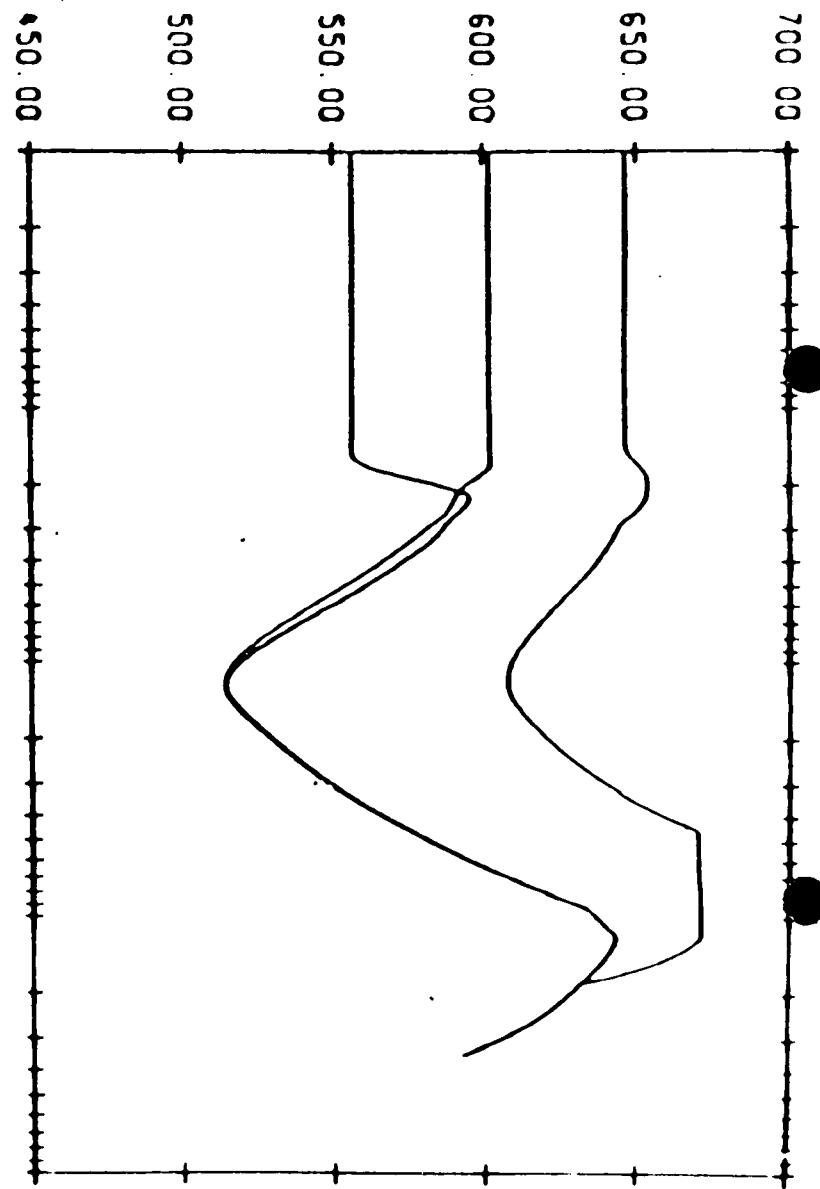
LOOP 2 TEMPERATURE(COLD HOT SAT)

DEGREES F



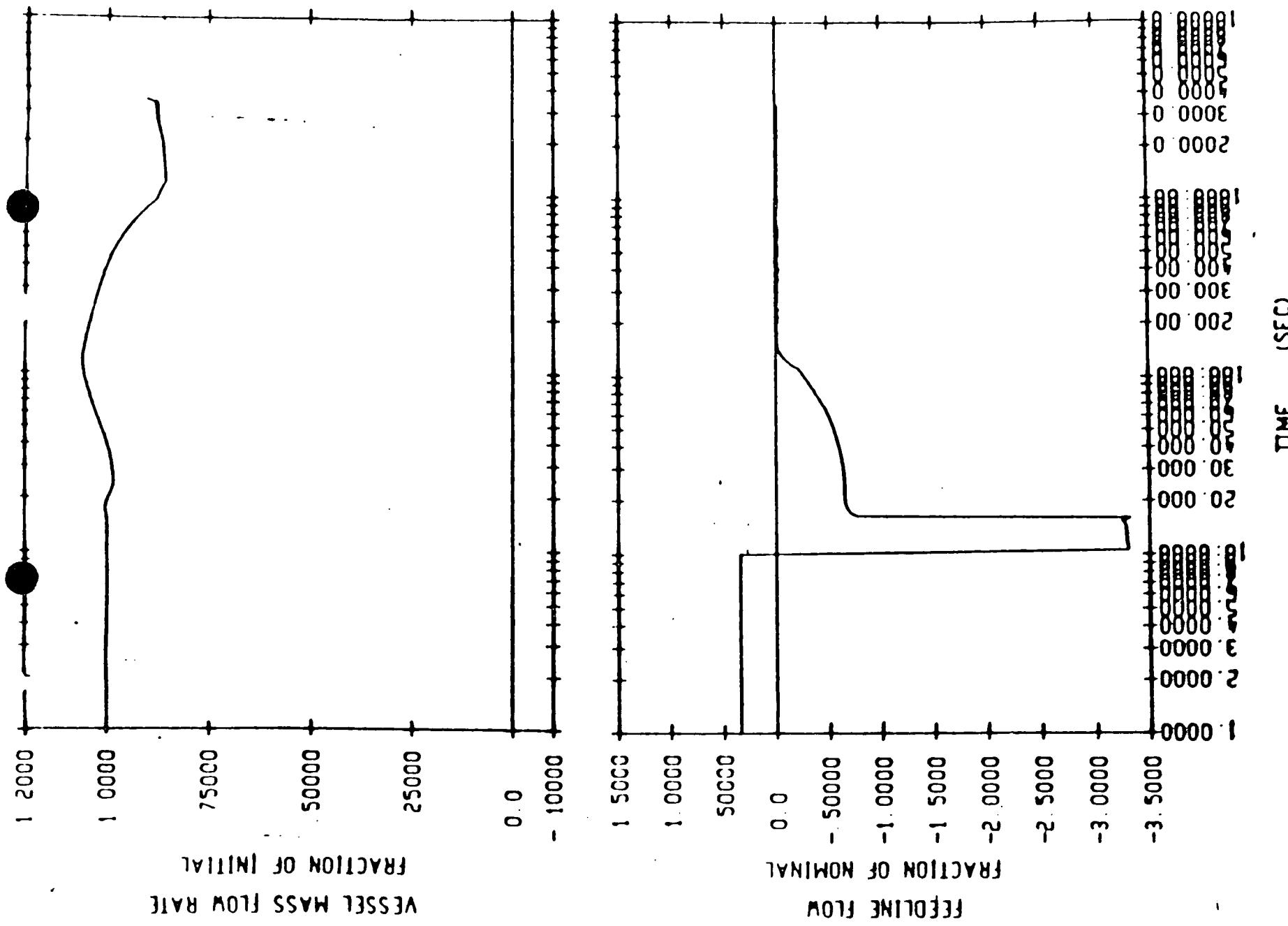
LOOP 1 TEMPERATURE(COLD HOT SAT)

DEGREES F

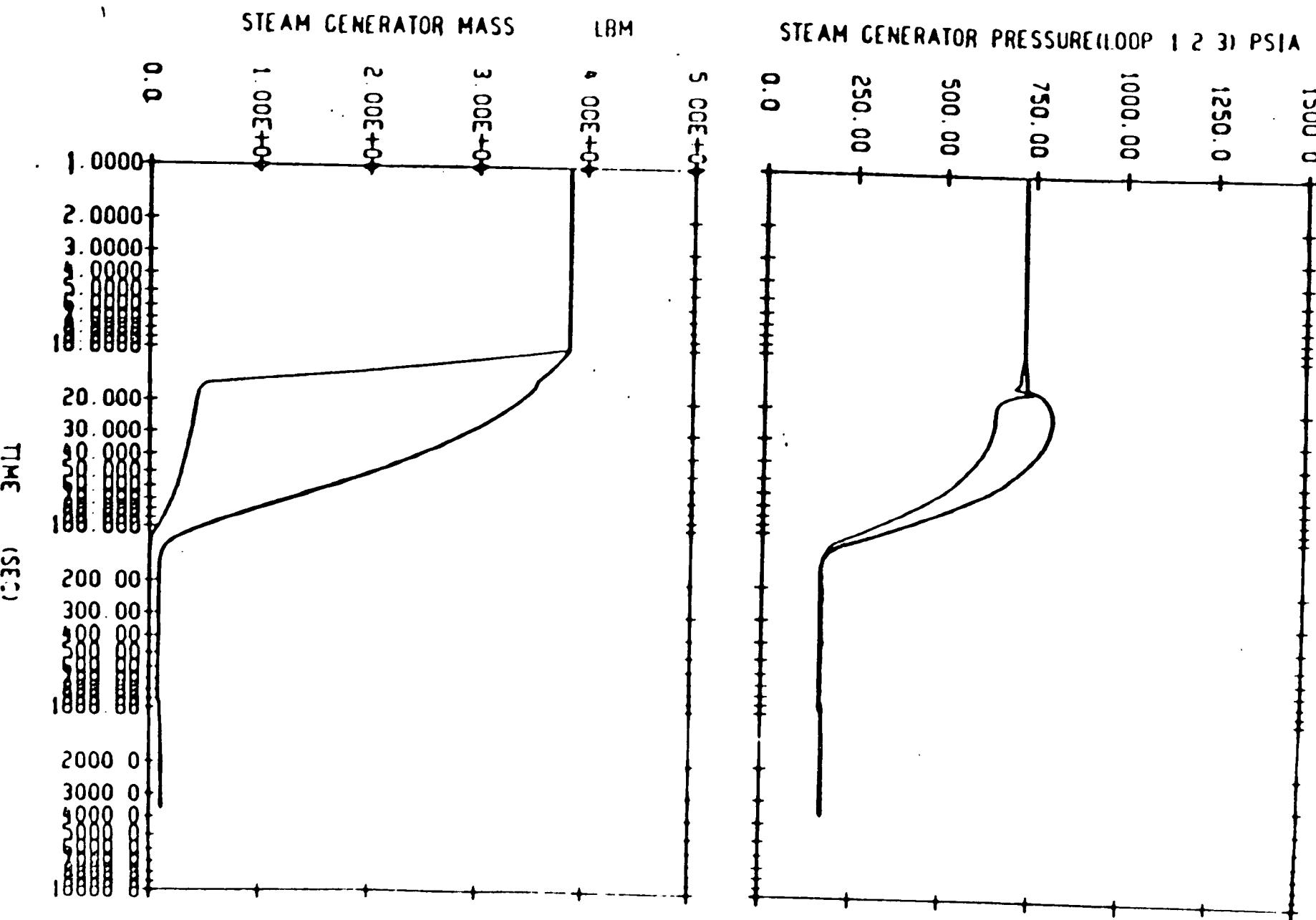


Feedline Rupture 250 gpm @ 15 min.

Feedline Rupture 250 gpm @ 15 min.



Feedline Rupture 250 gpm @ 15 min.



NRC Question No. 6

In the EPRI steam tests on a Crosby 3K6 Safety Valve, which was a test with a valve mounted on a short inlet pipe configuration, the valve opened and closed with 10-11% blowdown at the valve vendor's recommended ring settings. The rings were adjusted and blowdown was decreased to 4-5%, which meets the valve specification value of 4%. The submittal does not provide evidence, though, as to what the expected blowdown at the plant will be. It only states that "as installed" ring adjustments are expected to produce stable valve operation with 4-5% blowdown. Identify the "as installed" ring settings and determine the expected backpressure, since this too affects the blowdown. Present a calculation for an expected blowdown value based on the plant ring settings and backpressure. If the blowdown falls outside the 4-5% range, evaluate valve performance for the expected blowdown value.

SCE Response

The following represents the current San Onofre Unit 1 Safety Valve ring settings as developed by Crosby during Production Testing:

<u>Nozzle Ring</u>	<u>Adjusting Ring</u>
-6	-115

Note: The above ring settings are used on each safety valve and are measured from the highest locked position.

Valve ring settings developed by the Crosby Production test methods should have performance characteristics similar to those test valves, that were operated at "as-shipped" ring settings. This is true even though the ring setting numbers may differ between valves. This difference is due to part tolerance stack up within the individual valve and different ring movement per-notch for each valve size.

It is believed that insufficient information is available to extrapolate ring settings for the purpose of predicting blowdowns, especially for valves not tested. Therefore, it is not appropriate to predict valve performance based upon ring setting values.

NRC Question No. 7

Thermal expansion of the pressurizer and inlet piping to the valve would be expected to induce loading on the inlet flange at the time the valve is required to lift. Evaluate the effect that this loading may have on valve operability.

SCE Response

Maximum expected bending moment induced in the Safety Valve Flanges for the San Onofre Unit 1 Safety Valves has been calculated to be 44,994 inch-pounds due to deadweight and thermal effects. Since this value is much less than the 161,500 in-lb moment for the 3K6 test valve and the 298,750 in lb moment for the 6M6 test valve, the above loadings will not have a detrimental effect on valve operability.

NRC Question No. 8

Since the Crosby 3K26 valve was not specifically tested in the EPRI program, results from tests on the Crosby 3K6 and 6N8 valves were used for comparison. Flow rate data were only obtained from tests on the Crosby 3K6. Provide further information on how the data for the Crosby 3K6 valve was extrapolated to verify that the plant safety valve will pass its rated flow, particularly with the ring settings as adjusted at the plant.

SCE Response

As noted in Table 4.4 of EPRI Report NP-2770-LD, Volume 6, the Crosby 6M6 test valve achieved rated flow for each of the tests reported at 3 percent accumulation regardless of the ring setting used in the test. A review of EPRI Tables 4-3 and 4-4 in Volume 5 of EPRI Report NP-2770-LD reveals that for steam tests of the 3K6 valve where blowdown was measured to be less than 10 percent, flow rates of 119-122 percent of rated flow at 3 percent accumulation were reported. The EPRI tables indicate that lower than rated flows occurred at blowdowns greater than 15 percent for the 3K6 valve. No flow data was collected for the 6N8 valve. Crosby production tests for the San Onofre Unit 1 valves indicate 4-5 percent blowdown with the "as-shipped" ring settings. These are the ring settings currently installed on the San Onofre Unit 1 Safety Valves. This is within the range of both the 3K6 and 6M6 tests where rated flow was achieved; therefore, rated flow can be expected for the San Onofre Unit 1 Safety Valves.

NRC Question No. 9

The submittal indicates that a simultaneous opening of the two safety valves was assumed to produce the highest loading on the discharge piping system. The experience of EG&G Idaho indicates that the maximum forces are obtained when the sequence of opening reach the common header simultaneously. Provide additional justification that a simultaneous opening of the valves is adequate.

SCE Response

Considering the similar geometry and total length of the discharge piping system for both safety valves, the assumption of a simultaneous valve opening was made to assure that the initial pressure waves from the two valves opening reach the common header simultaneously. Examining the input for the time history analysis indicates that a total wave force of 9000 lb. was used. This is equal to the sum of wave forces from both valves opening with a 4500 lb. contribution from each valve. Therefore, the case of maximum forces on common header has been considered.

NRC Question No. 10

To calculate the time-dependent forcing functions acting on the piping a simplified graphical solution technique was applied. This technique is based on an ideal gas assumption. High pressure steam, though, is not an ideal gas. How well the technique applies to PWR conditions has not been demonstrated. The authors of the paper on this technique did compare calculations with test data for steam at 995 psia and obtained a good comparison. Provide further comparisons or verification to show the validity of this technique for PWR steam discharges.

SCE Response

Intermountain Technologies, Inc. (ITI), in Idaho Falls, was asked to evaluate independently the hydrodynamic loads due to valve actuation on the discharge piping using RELAP 5/MOD 1 Code. The analysis was for saturated steam upstream of the safety valves without assuming an ideal gas. The results of ITI's report are shown in Figures C-1 through C-7. These results demonstrate that SCE's results using simplified graphic solutions based on an ideal-gas assumption show good comparison in force characteristics and conservative data in force magnitudes. The discrepancy in Figure C-7 is judged not to be of any significant consequence because of the significant margins in all other pipe segments.

SO CAL EDISON PIPE LOAD ANALYSIS  
SEGMENT L11 FORCE 80LID=RELAP5,DR8HED=8CE 8C2XX

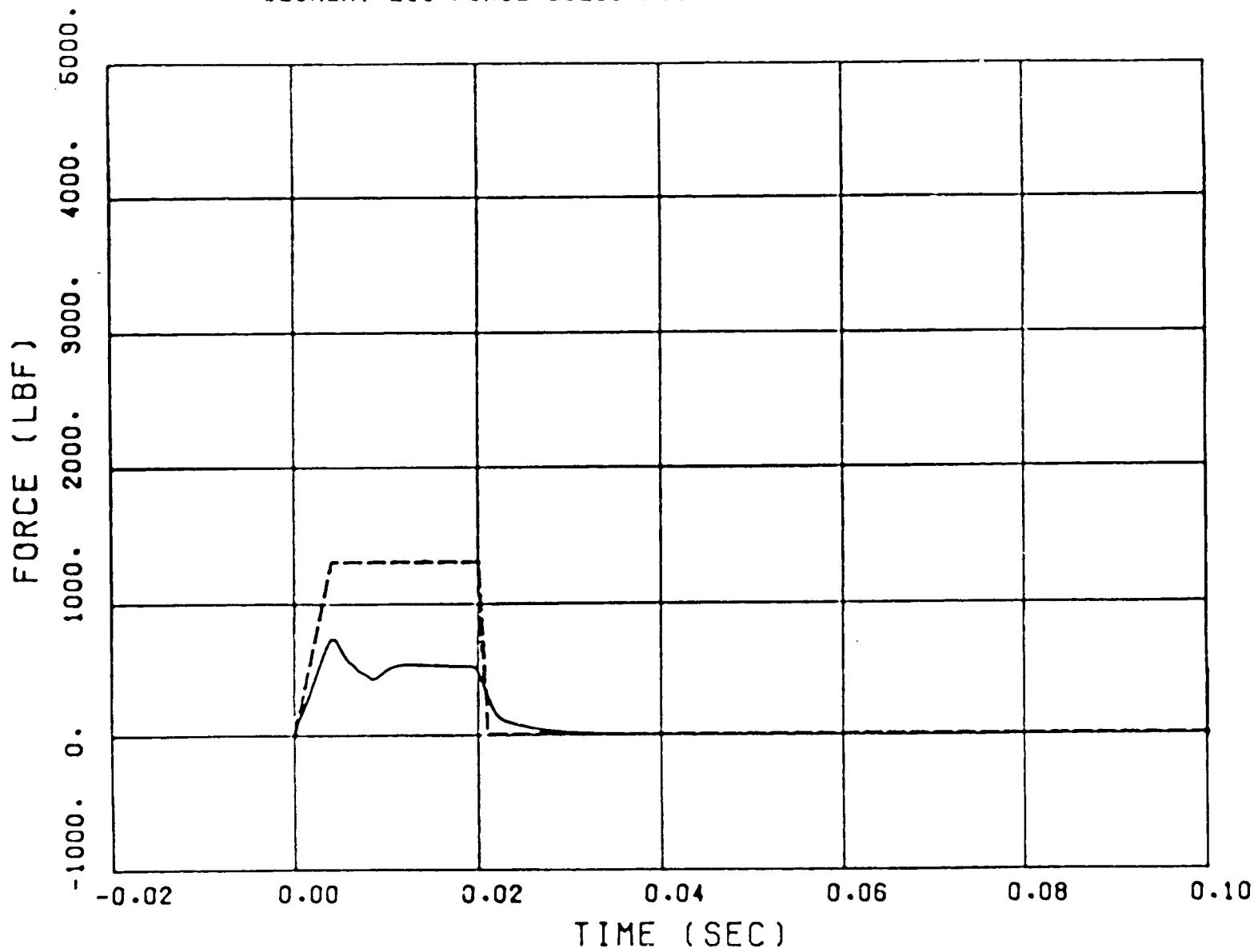


Figure C-1. Comparison between calculated hydrodynamic forcing functions on pipe segment L11 (RELAPS isentropic flow case versus SCE).

SO CAL EDISON PIPE LOAD ANALYSIS  
SEGMENT L12 FORCE 80LID=RELAPS,DASHED=8CE 8C2XX

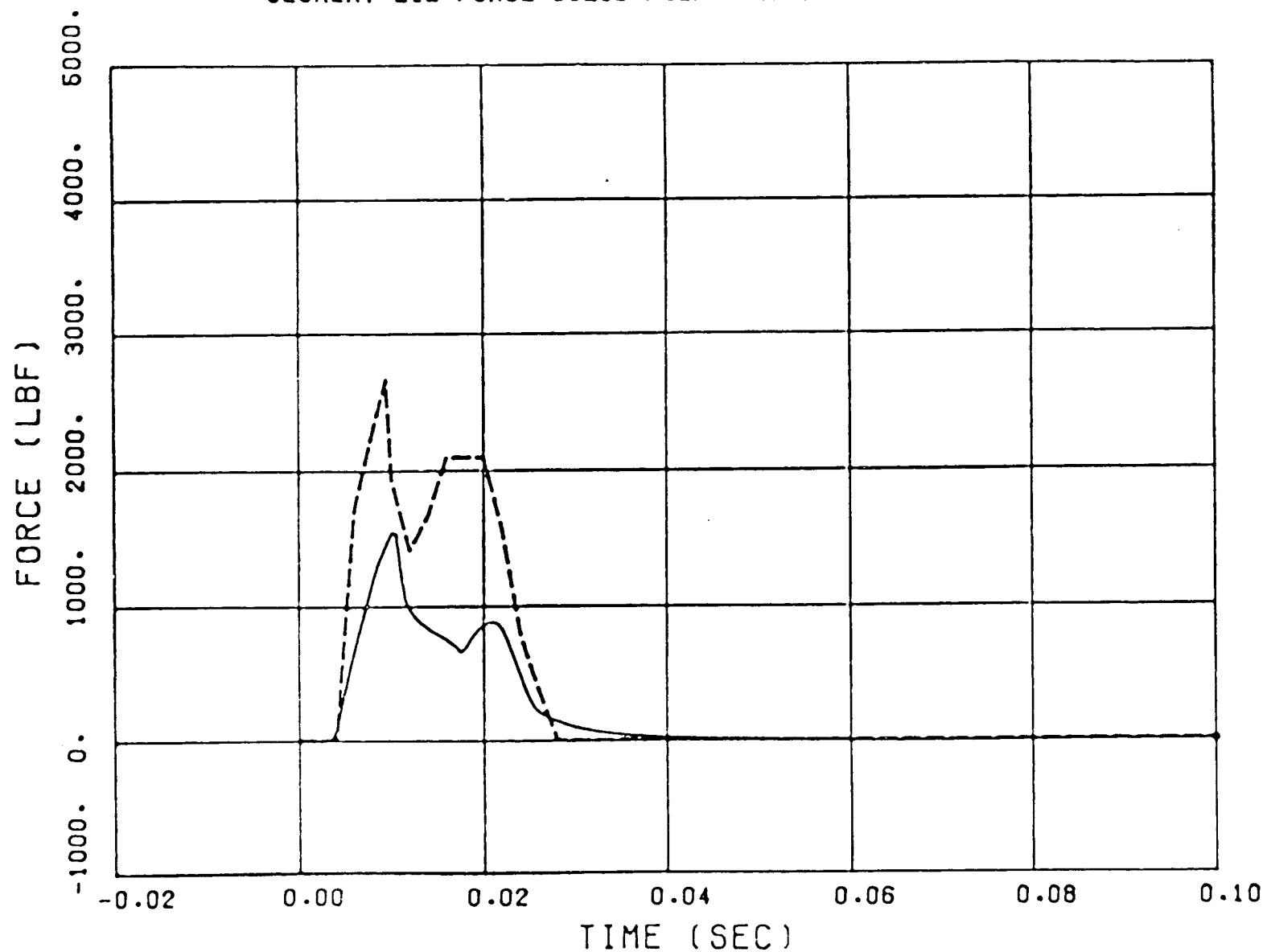


Figure C-2. Comparison between calculated hydrodynamic forcing functions on pipe segment L12 (RELAPS isentropic flow case versus SCE).

SO CAL EDISON PIPE LOAD ANALYSIS  
SEGMENT L13 FORCE 80OLID=RELAPS,0A8HED=SCE 8C2XX

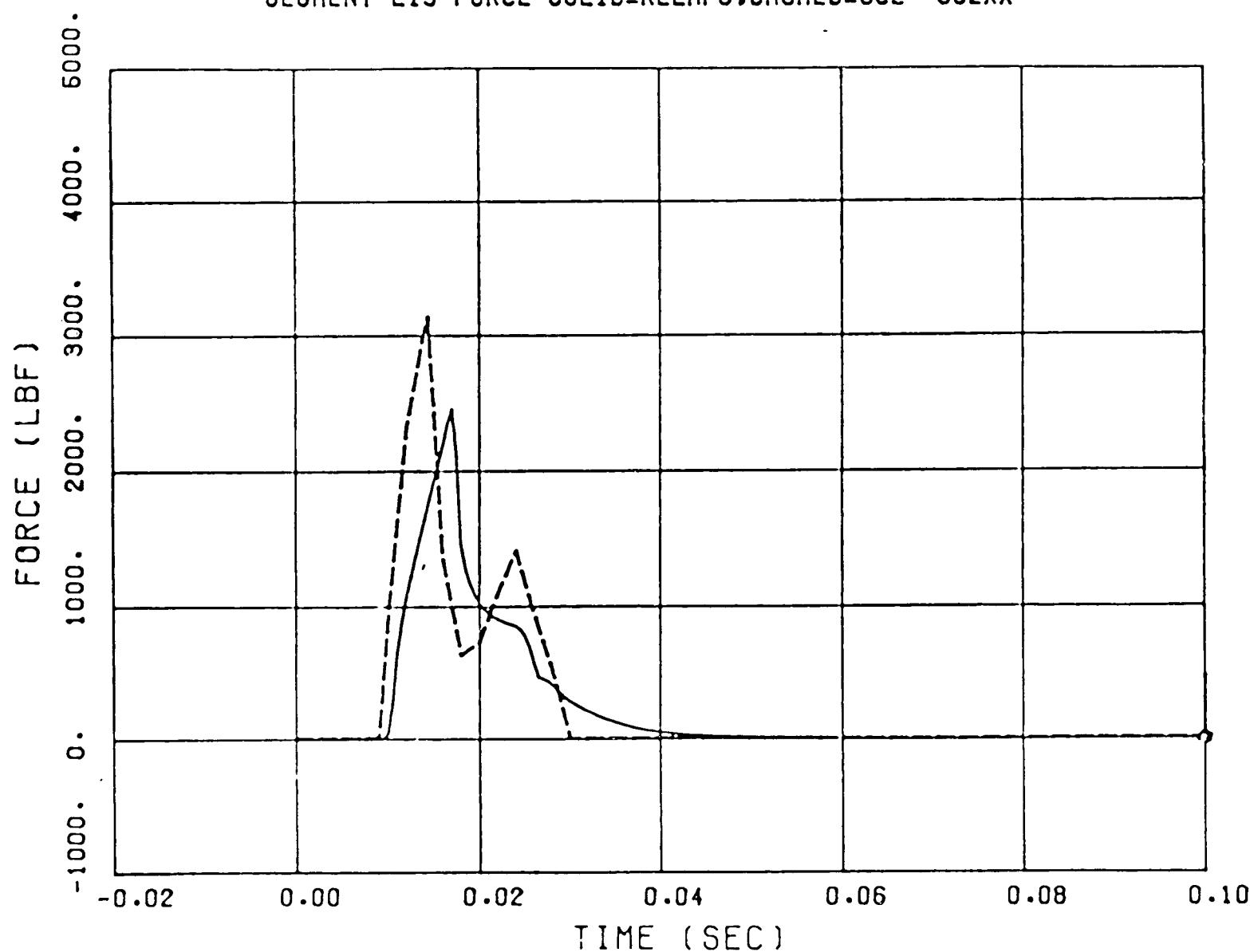


Figure C-3. Comparison between calculated hydrodynamic forcing functions on pipe segment L13 (RELAPS isentropic flow case versus SCE).

SO CAL EDISON PIPE LOAD ANALYSIS  
SEGMENT L14 FORCE 80LID=RELAPS.DA8HED=8CE 8C2XX

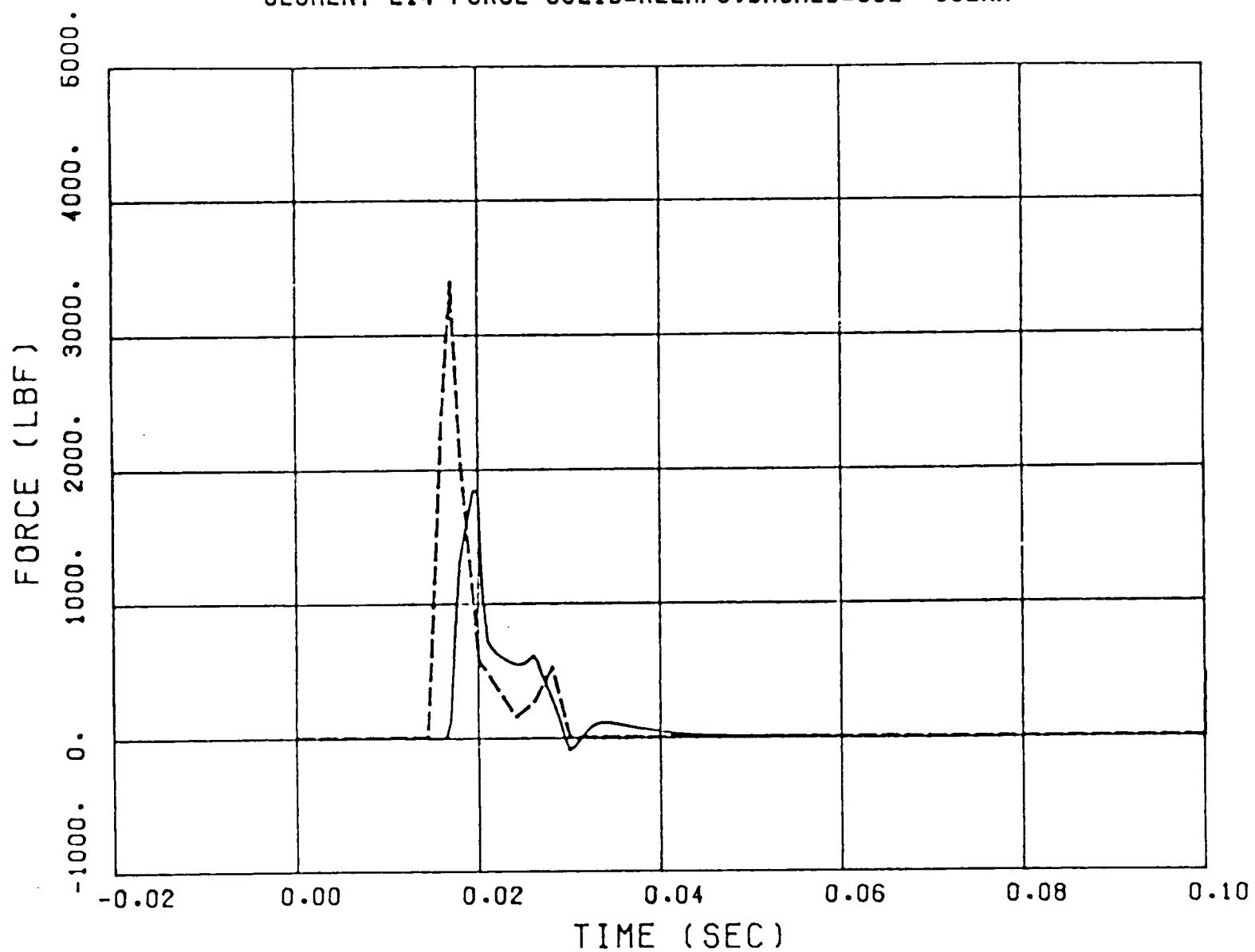


Figure C-4. Comparison between calculated hydrodynamic forcing functions on pipe segment L14 (RELAPS isentropic flow case versus SCE).

SO CAL EDISON PIPE LOAD ANALYSIS  
SEGMENT L16 FORCE 80LID=RELAP5,DA8HED=SCE 8C2XX

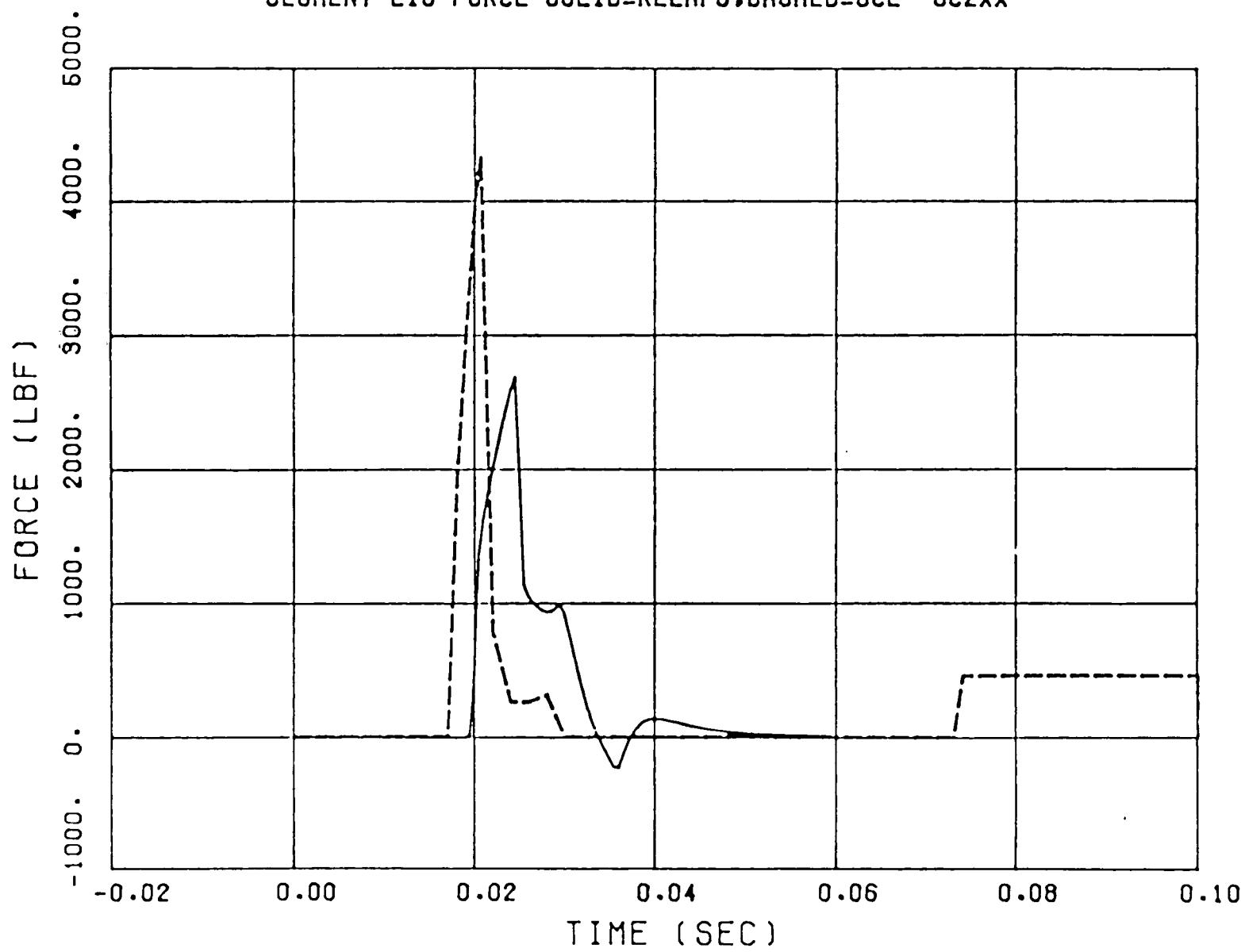


Figure C-5. Comparison between calculated hydrodynamic forcing functions on pipe segment L15 (RELAPS isentropic flow case versus SCE).

SO CAL EDISON PIPE LOAD ANALYSIS  
SEGMENT L16 FORCE SOLID=RELAP5,DR8HED=8CE 8C2XX

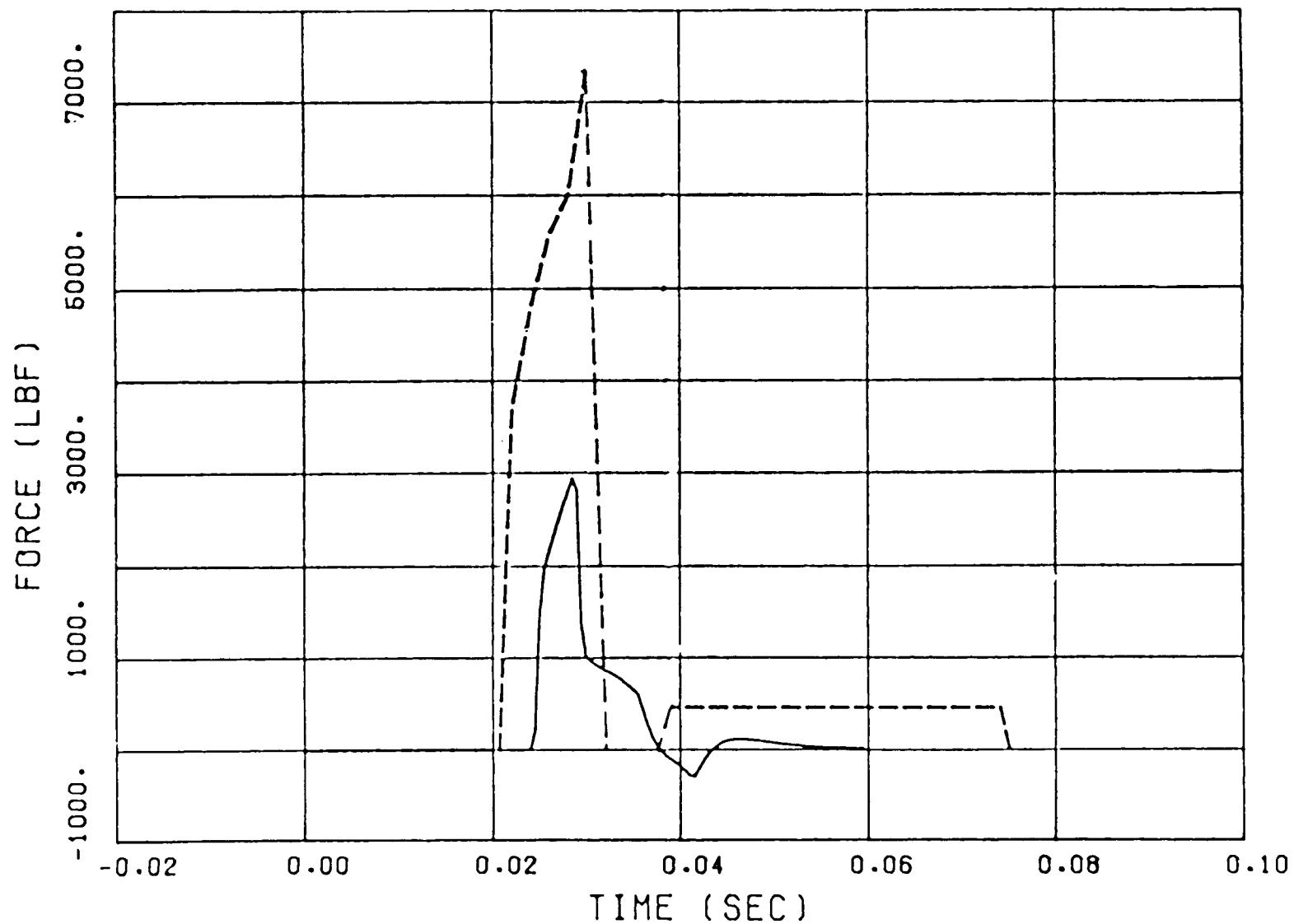


Figure C-6. Comparison between calculated hydrodynamic forcing functions on pipe segment L16 (RELAP5 isentropic flow case versus SCE).

SO CRL EDISON PIPE LOAD ANALYSIS  
SEGMENT L17 FORCE 80OLID=RELAPS.DA8HED=8CE 8C2XX

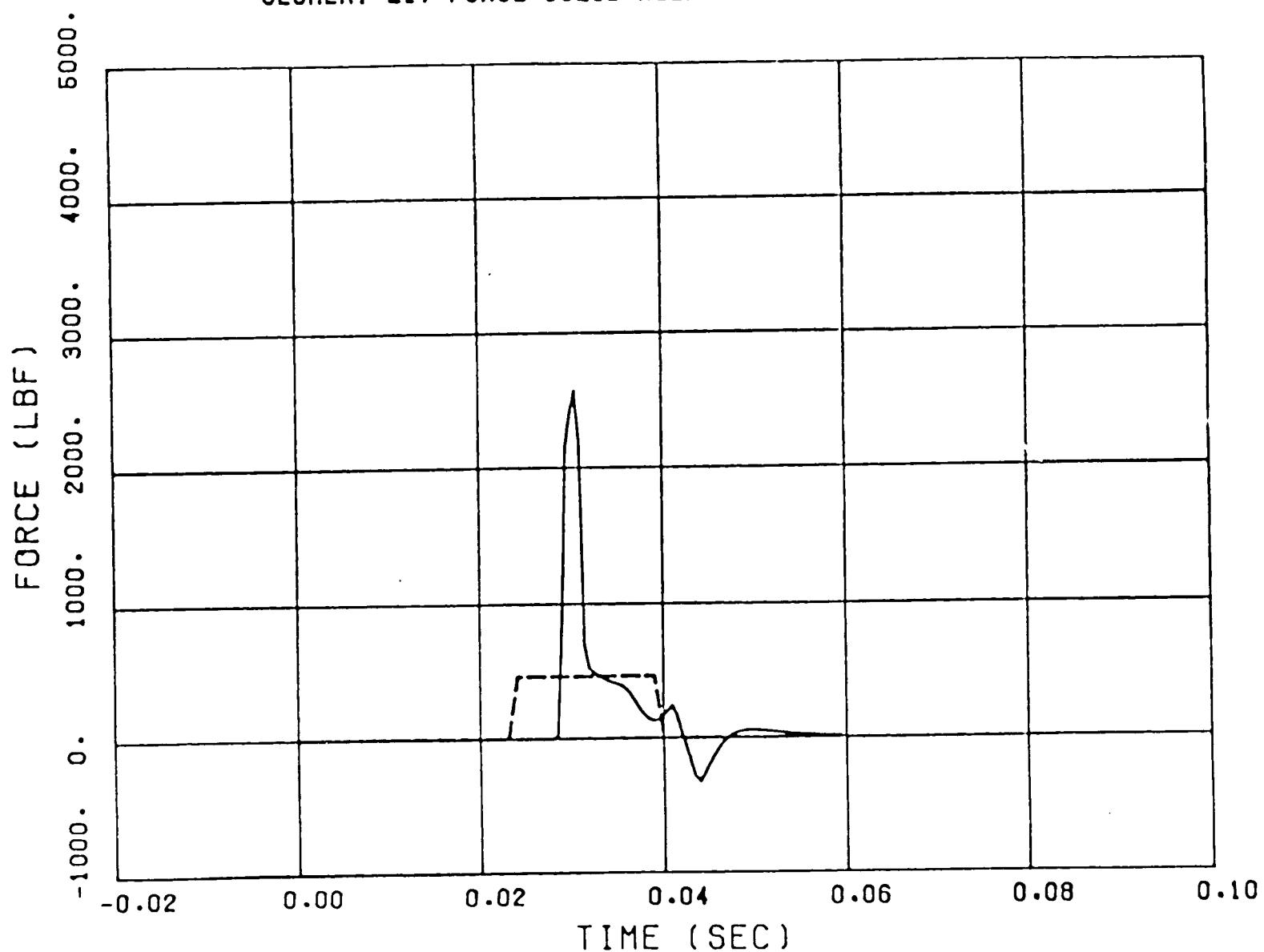


Figure C-7. Comparison between calculated hydrodynamic forcing functions on pipe segment L17 (RELAPS isentropic flow case versus SCE).

NRC Question No. 11

Differences between the ideal conditions assumed in the thermal hydraulic analysis and the conditions likely to be found in downstream piping, i.e., condensed liquid, were not addressed in the submittal. A sweeping along of this condensed liquid could result in larger forces on the piping. Thus, provide justification for the assumption of ideal conditions. Additionally, a valve opening time of 20 msec was selected for the analysis, while pop times on the order of 10 msec occurred in tests on the 3K6 and 6N8 valves. Provide justification for this difference.

SCE Response

Water vapor could condense in the downstream piping, since the ambient temperature in the containment can be as high as 120°F corresponding to a saturation pressure of approximately 2 psig, which is below the discharge piping system backpressure of 3 psig. However, judging from piping routing of the 10" common header, it is very unlikely for any significant amount of condensed liquid to be accumulated in the pipe because of relative shortness of horizontal segment. Further, due to the large upward thermal movement of the pressurizer nozzles, approximately 1.93", the 6" piping downstream of the valve tends to be sloping toward the header. Any significant amount of condensed liquid would be drained down to the pressure relief tank. Therefore, the water entrainment phenomenon is judged to be insignificant.

At the time of the analysis as presented in the submittal, a literature survey was conducted and the fastest opening time was found to be 20 msec. As of today, EPRI test data on Crosby 3K26 valve is not available. It is judged that conservatisms in the analysis, as evidenced in ITI's Relap V results should envelop any larger load due to a shorter opening time.

NRC Question No. 12

Some of the information needed to evaluate the structural analysis was not supplied in the submittal. So as to provide information on the stiffness values used to model the supports, the calculated values for loads on the supports, the time step used in forcing functions and similar information, provide a computer printout of input and output for the Locked Rotor saturated steam case analysis.

SCE Response

One copy of the computer printout of input and output for the Locked Rotor saturated steam case analysis is attached herewith. For easy reference, parameters of interest are summarized in the following:

1. Stiffness values of supports

$10^{10}$  lb/in      translational

$10^{10}$  in-lb/rad    rotational

2. Calculated values of support load

Please see Table 3 in Enclosure 2 of the December 2, 1982 submittal.

3. Time step used in forcing functions

0.0001 sec. was specified for integration steps.

NRC Question No. 13

The submittal does not make clear whether flexibility at the connection locations to the pressurizer and discharge tanks was considered in the development of the structural model. Explain how this flexibility was treated in the piping model.

SCE Response

The connection locations to the pressurizer and discharge tank were treated in the piping model as six springs with stiffness values of  $10^{10}$  lb/in for three translational directions and  $10^{10}$  in-lb/rad for three rotational directions.

NRC Question No. 14

Section 3.1, pg. C-8, of the submittal states that an opening of the PORV's does not require analysis because the PORV opening time is much slower than that of the safety valves. Thus, the safety/relief valve system was not analyzed for a PORV opening, and the PORV piping was ignored in the structural analysis. The PORV piping is, however, constructed of smaller size piping and is arranged in a different configuration from that of the safety valve piping. Additionally, the PORV piping ties into a header that is common with the safety valve piping, creating a complex interactive piping system. Thus, it is not obvious on the basis of relative opening times alone that the PORV piping can be ignored. Provide a more thorough evaluation of the safety of the PORV piping.

SCE Response

The PORV piping has been combined with the safety valve piping for the time history structural analysis since the submitting of SCE report on December 2, 1982. In addition to the valve actuation analysis, loading cases of seismic, thermal and weight have also been analyzed. Results of code criterion evaluation are listed in the attached tables.

ENGINEERING DEPARTMENT  
CALCULATION SHEET

SUBJECT S.R. VALVE DISCHARGE PIPING (SONGS-1) DESIGN NO. DC

CALCULATION NO. DC

JO NO. 10 NO. MADE BY J. CHAN DATE 5/28/85 CHECKED BY M.G.M. DATE 5/29/85

Table PORV DISCHARGE PIPING STRESS EVALUATION

HOSEL PT.	TYPE OF COMPONENT	PRESSURE STRESS $S_{LP}$	DEADWEIGHT LCWL, $M_A$ (IN-#)	OCCASIONAL LOAD (IN-#) $M_B$		$S_{OL} = \frac{S_{LP} + .75 L \sqrt{\frac{M_A + M_B}{2}}}{2}$ PSI	REMARKS
				OEE ( $M_{R1}$ )	VALVE ACTUATION ( $M_{R2}$ )		
100	4" NTEE SCM 40	2373	278	2129	9717	10006	5857 SOL 2 SALLOW ∴ SATISFACTORY
102	4" BW. ELL SCM 40	2373	490	2508	7374	7788	4951
105	"	2373	697	842	9737	9773	5634
109	"	2373	546	904	12219	12252	6359
51	10" REINF. TEE SCM 40 t <sub>e</sub> = 3/8	2373	2547	4335	33205	33486	25019
78	2" C/W TEE SCM 160	4315	802	5427	3157	6278	11546
111	2" BW. JT. SCM 160	4315	2113	4049	1604	4355	10921
117	2" BW JT SCM 40	1927	2615	4254	2349	4859	15249
119	4" X 2" R.L.D. SCM 40	1927	2135	3468	2728	4412	13597

● ENGINEERING DEPARTMENT  
CALCULATION SHEET

SUBJECT S.B. VALVE DISCHARGE PIPING (SONGS-1)

DESIGN CALCULATION NO. DC

10 NO. MADE BY A. Chet DATE 5/28/85 CMW OR M/S/AM- DATE 5/1/85

NO/AL PT	TYPE OF COMPONENT	PRESSURE STRESS CLD	SEA. WICHT LCL, $M_A$ (IN-#)	OCCASIONAL LOAD (IN-#) $M_B$			$SOL = \frac{S_{sp} + .75L}{P_B}$	REMARKS
				OBE ( $M_{R1}$ )	VALVE ACTUATION ( $M_{R2}$ )	$M_B = \sqrt{M_{R1}^2 + M_{R2}^2}$		
133	3" FRES. NO.3. SCA 160	4994	3098	14699	4737	15442	5000	$SOL < SALLOW$ $\therefore$ Satisfactory
75	2" SW REDUCING COUPLING SCA 160	4315	1119	8813	836	8852	14499	"
76	2" SW TEE SCA 160	4315	570	8832	959	3950	8931	"
76	"	4315	868	3845	1304	4079	9338	"
78	"	4315	1610	4327	2005	4768	10829	"
80	2" SW ELECTN SCA 160	4315	677	3639	1581	3967	9058	"
83	2" BN JT. SCA 160	4315	2320	6032	796	6084	12899	"
87	"	4315	1661	2794	920	2941	9015	"
89	2" BN JT. SCA 40	1927	779	1673	1160	2035	6950	"
91	4x2" RED. SCA 40	1927	144	2177	1570	2684	6977	"
97	4" BN ELL SCA 40	2373	96	1538	3090	3451	3989	"

$$SALLOW = 1.8 F_{sh}$$

$$1.8 (1676) = 3016.8 \text{ PSI FOR A312 TP 316}$$

$$= 1.8 (1500) = 2700 \text{ PSI FOR A106 CR. B}$$

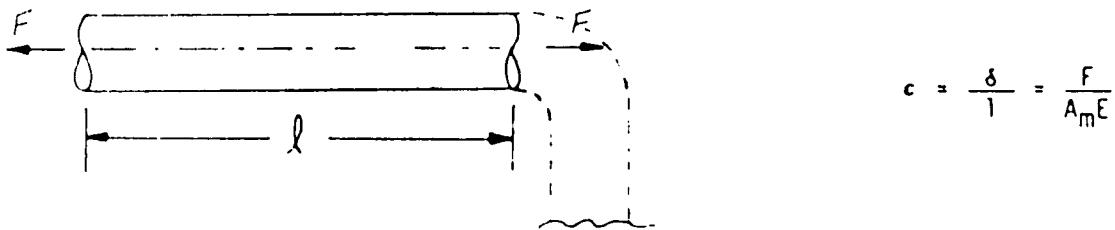
NRC Question No. 15

In the structural analysis only a net "wave" force was applied to each bounded segment of straight pipe. Applying only a wave force on a pipe segment, however, ignores axial extension that is caused by opposing blowdown forces acting at the ends of the segment. This axial extension induces bending moments on pipe segments adjacent to the bounded segment in question. Explain how such axial extension was accounted for in the analysis.

SCE Response

Based on experience in piping design, stresses and moments induced by "axial extension" on the bounded pipe segment or its adjacent segments are small when compared to the same quantities obtained from pipe flexibility analysis which treats piping segments as structural elements.

For clarity, the maximum pipe strain caused by axial extension is calculated for both 6" piping and 10" common header.



Where

$\delta$  = pipe deflection due to the axial extension, in

$l$  = length of piping, in

$F$  = axial force,  $lb_f$

$A_m$  = cross section of piping, in<sup>2</sup>

$E$  = Young's modulus of the piping material,  $lb_f/in^2$

From computer output, the largest axial piping reaction is at nodes 72 and 65 with values of  $16,330\ lb$  and  $13,520\ lb$ , respectively. The corresponding pipe strains are:

$$\epsilon = \frac{16300}{5.58 \times 27.9 \times 10^6}$$
$$= 0.0105\% \quad \text{for 6" pipe}$$
$$\epsilon = \frac{13520}{11.91 \times 27.9 \times 10^6}$$
$$= 0.0041\% \quad \text{for 10" pipe}$$

These values for strain and resulting pipe deflections are small and acceptable.

NRC Question No. 16

The submittal does not discuss the method used for modeling the safety valves. Of particular concern is the portion of the valve that lies off the main axis of the piping. The flexibility of this part of the valve structure should be accounted for in the model if its natural frequencies could potentially be excited by piping or support motion. Describe the methods used to represent the valves in the piping model.

SCE Response

The method used to represent the valves in the piping model is to consider each as a piping segment with the same diameter and wall thickness as its connecting pipe. In this way, the flexibility of the whole valve is accounted for in the piping model. Therefore, its natural frequencies can be excited by piping motion and conservative results are obtained for dynamic loadings such as valve-actuation transients and seismic events.

NRC Question No. 17

Based on diagrams of the structural model that were provided in the submittal, the spacing between lumped masses appears to be far enough that the higher frequency response of the piping system will be precluded from the solution. Thus, though the solution time step (0.0001 s) is quite fine and would allow for contributions from the higher modes, the spacing between lumped masses would seem to eliminate these contributions. Explain how the nodal spacing in the piping model was established.

SCE Response

The nodal spacing in the piping was established based on the consideration that the entire response of the piping system at all frequencies under 33 Hertz should be included in the solution. It is noted that the figures provided with the December 2, 1982 submittal did not accurately indicate the linear distance between the nodes. However, the largest nodal spacing was 6.12' between nodes 46 and 73 for the 6" pipe and 4.90' between nodes 67 and 68 for the 10" common header. This nodal spacing is consistent with the above statement that the entire response of the piping system should be observed.

NRC Question No. 18

The submittal does not discuss the results of the stress analysis on the piping upstream of the safety valves. Identify the governing Code, the applicable criteria from the Code, and the load combinations used in the analysis of this portion of the piping. Also, provide an evaluation of the stresses relative to the Code requirements.

SCE Response

NUREG-0737 requested all operating plant owners to verify the designed performance of pressurizer safety/relief valves and the integrity of discharge piping. As concluded in several workshop meetings, all utility companies understood that the upstream piping was not in the scope. Therefore, no results of the stress analysis was discussed in the submittal. However, in analyzing the discharge piping, upstream piping was also included because a complete problem in piping stress analysis is from anchor (pressurizer nozzle) to anchor (relief tank nozzle).

The whole system was analyzed using ANSI B31.1 Code based on 10 CFR, Part 50. Load combinations consist of weight, thermal expansion, seismic and valve actuation loadings. A summary table of the upstream piping is attached.

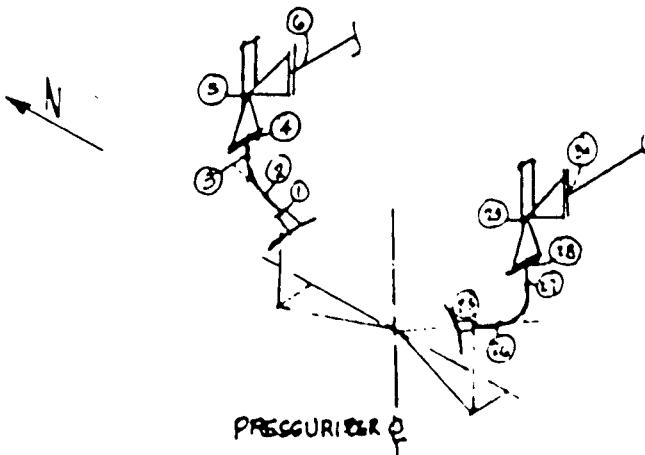
ENGINEERING DEPARTMENT  
CALCULATION SHEET

SUBJECT: G/R VALVE DISCHARGE PIPING (SONGS-1) DESIGN CALCULATION NO. DC

I.D. NO. 7704 MADE BY A Ch DATE 5/22/85 CHK'D BY ACAO DATE 5/22/85

TABLE - R-1  
VALVE-ACTUATION LOADS ON S/R VALVE UPSTREAM PIPING

NODAL PT.	PIPING REACTIONS						REMARKS	
	FORCES (LBS.)		MOMENTS (IN-LBS.)					
	F <sub>X</sub> IAL	F <sub>Y</sub>	F <sub>Z</sub>	M <sub>TORQUE</sub>	M <sub>y</sub>	M <sub>Z</sub>		
1	1295	2559	2714	16430	25780	21460	3" PRESSURIZER NOZ.	
2	1295	2559	2714	16430	19000	18040		
3	2681	2862	6253	6566	13440	16400		
4	2681	2823	625	6566	8722	9661	3" INLET FLANGE	
5	2679	2823	572	6566	5151	36130		
6	2761	2676	463	5151	3840	20670	6" OUTLET FLANGE	
25	808	1167	2281	38510	31400	22950	3" PRESSURIZER NOZ.	
26	807	1167	2281	38510	25620	19540		
27	818	2355	957	14790	25760	32850		
28	813	2299	853	14790	21420	24440	3" INLET FLANGE	
29	813	2299	853	14790	18670	28100		
30	2193	808	633	18670	12680	24990	6" OUTLET FLANGE	



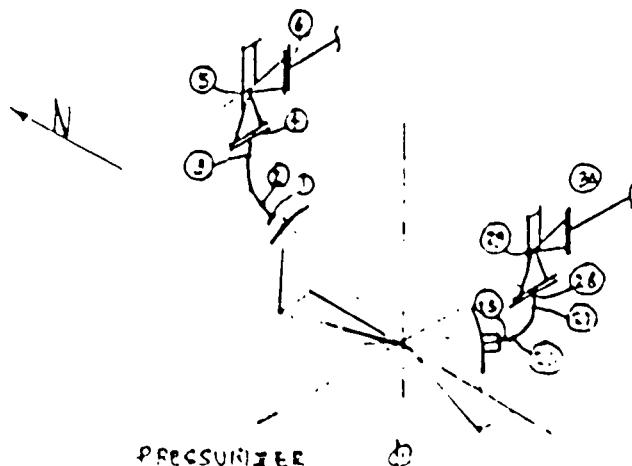
ENGINEERING DEPARTMENT  
CALCULATION SHEET

SUBJECT: S/R VALVE DISCHARGE PIPING (SONGS-1) DESIGN SHEETS  
 I.O. NO. 7764 MADE BY D. Chinn DATE 5/24/85 CHK. BY M. Gora DATE 5/24/85

TABLE - R-2

DEADWEIGHT LOADS ON S/R VALVE UPSTREAM PIPING

NODAL PT	PIPING REACTIONS						REMARKS	
	FORCES (LBS)			MOMENTS (IN-LBS)				
	FAXIAL	F <sub>Y</sub>	F <sub>Z</sub>	M <sub>TORSION</sub>	M <sub>Y</sub>	M <sub>Z</sub>		
1	-269	-234	88	464	-793	-806	3" PRESSURIZER NO2.	
2	-266	-281	88	464	-574	-224		
3	-346	25	88	38	-455	158		
4	-332	-91	-4	38	-43	844	3" INLET FLANGE	
5	-816	-91	-4	38	-76	1197		
6	91	179	-4	76	2	-517	6" OUTLET FLANGE	
25	-413	-398	14	1025	-121	-765	3" PRESSURIZER NO3	
26	-410	-395	14	1025	-80	425		
27	-563	11	14	686	-738	1138		
28	-549	10	-14	686	1180	261	3" INLET FLANGE	
29	-533	10	-14	686	1045	170		
30	-10	-37	-14	-1045	542	624	6" OUTLET FLANGE	

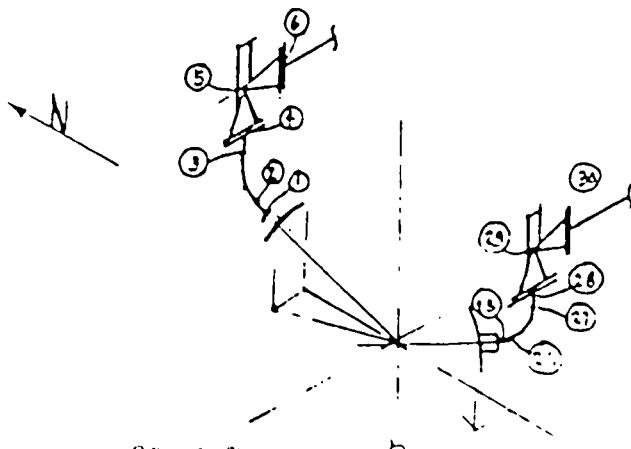


ENGINEERING DEPARTMENT  
CALCULATION SHEET

SUBJECT: S/R VALVE DISCHARGE PIPING (SONGS-1) DESIGN  
 J.O. NO. 7764 MADE BY d che DATE 5/24/85 CHECKED BY KO GAN DATE 6/24/85

TABLE - R-3  
THERMAL EXP (COLD) LOADS ON S/R VALVE UPSTREAM PIPING

NODAL PT	PIPING REACTIONS						REMARKS
	F AXIAL	F Y	F Z	M TORSION	M Y	M Z	
1	-1120	-554	792	18357	-32152	6727	3" PRESSURIZER NOZ.
2	-1120	-554	792	18357	-30171	8113	
3	-1188	404	793	-7315	-31796	8368	
4	-1188	-879	139	-7316	3121	-24852	3" INLET FLANGE
5	-1188	-879	139	-7316	4427	-16614	
6	879	-1188	139	-4427	-5923	-4733	6" OUTLET FLANGE
25	-1283	-300	-174	-12619	33022	35842	3" PRESSURIZER NOZ
26	-1283	-300	-174	-12619	32500	36743	
27	-1120	695	-174	13838	31355	36000	
28	-1120	-380	-607	13836	19042	-37493	3" INLET FLANGE
29	-1120	-980	-607	13836	13349	-33929	
30	380	-1120	-607	-13349	7763	-22733	6" OUTLET FLANGE



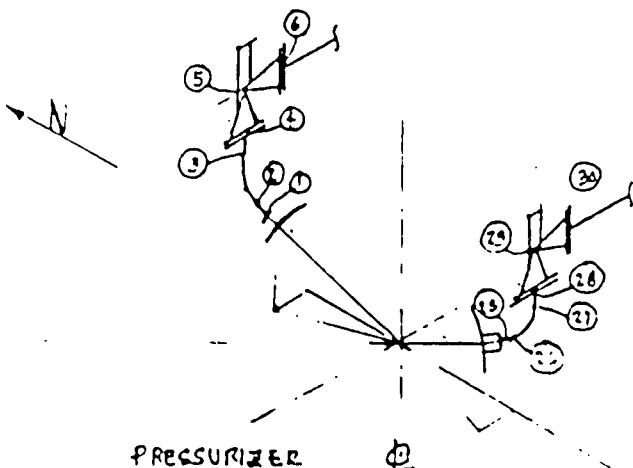
ENGINEERING DEPARTMENT  
CALCULATION SHEET

SUBJECT: S/R VALVE DISCHARGE PIPING (SONGS-1) DESIGN SHEET NO. DC \_\_\_\_\_  
 J.O. NO. 7764 MADE BY d chen DATE 5/24/85 CHK. BY Atkam DATE 5/24/85

TABLE - R-4

THERMAL EXP (HOT) LOADS ON S/R VALVE UPSTREAM PIPING

NODAL PT	PIPING REACTIONS						REMARKS
	FAXIAL	F <sub>y</sub>	F <sub>z</sub>	M <sub>TORSION</sub>	M <sub>y</sub>	M <sub>z</sub>	
1	-751	222	-624	-15970	-19240	30259	3" PRESSURIZER NO2.
2	-751	222	-624	-15970	-20800	-2774	
3	-374	689	-624	-26828	-5415	28000	
4	-374	381	848	-26828	-17266	-17248	3" INLET FLANGE
5	-374	381	848	-26828	-9315	-20817	
6	-381	-374	848	9315	-18347	-17078	6" OUTLET FLANGE
25	248	-1007	204	-30959	3285	-16269	3" PRESSURIZER NO2
26	248	-1007	.204	-30959	-13248	-5376	
27	-537	-887	205	-18859	25301	-9723	
28	-536	469	781	-18860	-9977	-25312	3" INLET FLANGE
29	-536	469	781	-18860	-2056	-29706	
30	-469	-536	781	2656	-11051	-29706	6" OUTLET FLANGE



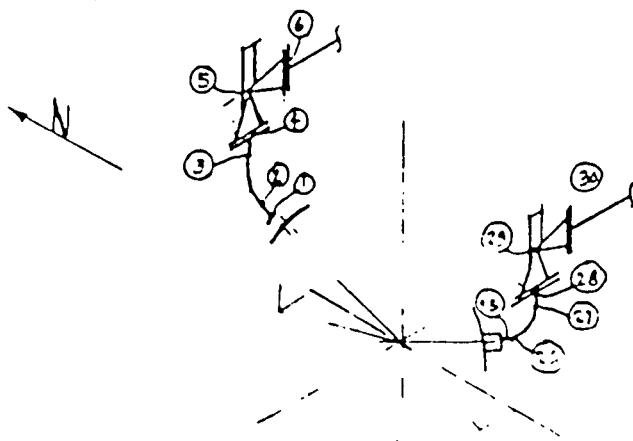
ENGINEERING DEPARTMENT  
CALCULATION SHEET

SUBJECT S/R VALVE DISCHARGE PIPING (SONGS-1) DESIGN SHEET NO. DC \_\_\_\_\_  
 J.O. NO. 7764 MADE BY J. Chen DATE 5/24/85 CHK. BY A. Garcia DATE 5/24/85

TABLE - R-5

SEISMIC (OBE) LOADS ON S/R VALVE UPSTREAM PIPING

NODAL PT	PIPING REACTIONS						REMARKS	
	FORCES (LBS)			MOMENTS (IN-LBS)				
	FAXIAL	F <sub>X</sub>	F <sub>Z</sub>	M <sub>TORSION</sub>	M <sub>X</sub>	M <sub>Z</sub>		
1	178	203	178	1733	2213	4384	3" PRESSURIZER NOZ.	
2	178	203	177	1733	1777	3880		
3	79	188	269	410	3266	1902		
4	79	186	266	410	911	479	3" INLET FLANGE	
5	57	72	32	1697	410	1557		
6	55	74	21	1697	287	876	6" OUTLET FLANGE	
25	167	231	181	2034	2557	4688	3" PRESSURIZER NOZ.	
26	167	231	181	2034	2017	4012		
27	51	191	274	662	3239	2130		
28	50	188	270	662	916	833	3" INLET FLANGE	
29	36	39	24	1908	662	1662		
30	29	39	26	1908	594	1511	6" OUTLET FLANGE	



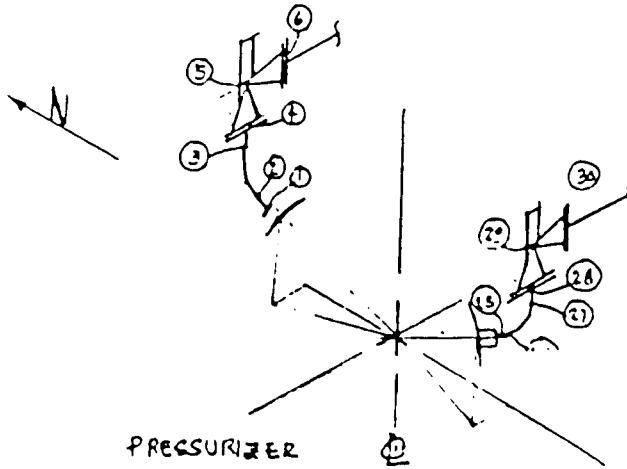
ENGINEERING DEPARTMENT  
CALCULATION SHEET

SUBJECT: S/R VALVE DISCHARGE PIPING (SONGS-1) DESIGN SHEET NO. DC  
 J.O. NO. 7764 MADE BY d chen DATE 5/24/85 CHK. BY KOTAM DATE 5/24/85

TABLE - R-6

SEISMIC (DBE) LOADS ON S/R VALVE UPSTREAM PIPING

NODAL PT	PIPING REACTIONS						REMARKS	
	FORCES (LBS)		MOMENTS (IN-LBS)					
	FAXIAL	F <sub>y</sub>	F <sub>z</sub>	M <sub>TORSION</sub>	M <sub>y</sub>	M <sub>z</sub>		
1	1062	1217	975	11520	12940	27510	3" PRESSURIZER NO2	
2	1062	1216	975	11520	10500	24630		
3	456	1007	1612	3741	20840	11450		
4	456	1007	1612	3741	6726	3514	3" INLET FLANGE	
5	470	420	228	9310	3741	7562		
6	459	432	163	931	275	365	6" OUTLET FLANGE	
25	1274	1782	1267	13880	18570	35290	3" PRESSURIZER NO2	
26	1273	1782	1267	13880	14890	29950		
27	361	1320	2103	6128	24540	14590		
28	359	1311	2086	6128	6618	6036	3" INLET FLANGE	
29	251	266	214	15240	6128	11940		
30	200	265	248	15240	4984	1096	6" OUTLET FLANGE	



NRC Question No. 19

For piping downstream of the safety valves, the operating basis earthquake (OBE) loading was not combined with the dynamic loads from the safety or relief valve discharge. The rationale for not combining these loads was that the downstream piping is non-safety related and that the probability of a seismic event coinciding with a safety valve actuation is extremely low. According to the requirements of the governing ANSI B31.1 Code, however, the discharge transient loads and the earthquake loads should both be included in the stress calculation equation (Equation 12). Thus, though a simultaneous occurrence of these loads is not highly probable, they would be combined to meet governing criteria. In addition, a load combination in which safe shutdown earthquake (SSE) loads are combined with safety/relief valve fluid transient loads (as a faulted condition in Westinghouse report WCAP 10105) for the upstream piping should be considered. It is not clear whether the seismic loads mentioned in the submittal pertain to a safe shutdown earthquake. Present an assessment of the downstream piping whereby OBE loads are included. Verify that the load combinations on the upstream piping include a case in which SSE loads are combined with safety/relief valve loads. Identify the allowable stress limits used for this case.

SCE Response

Discharge piping system of the safety and relief valves in San Onofre Unit 1 is a closed system. It is an industry practice that the moments produced by valve actuation are not combined with seismic loading for a system of this nature.

However, responsive to your request, the results of combining OBE and valve actuation loadings are presented in the tables attached. The criterion as specified in Equation (12) of the governing ANSI B31.1 Code has been met.

Results of load combination on the upstream piping including the case in which SSE loads are combined with S/R valve loads are also attached. An allowable stress of  $2.4s_h$  is used for this case.

**CALCULATIONS SHEET**

ENGINEERING DEPARTMENT

S/R. VALVE DISCHARGE PIPING (SONG-1)

PROBLEM CALCULATION NO. DC

I.O. NO.

7764

MADE BY A. ECACARIO

DATE 5/20/85

CM. BY

B/C

DATE 5/21/85

Table Pressurizer S/R V Discharge Piping

MODE PT.	TYPE OF COMPONENT	SLP PSI	M <sub>A</sub> (WEIGHT) IN-LBS.	OBE MOMENTS (IN-#)				VALVE ACTUATION MOMENTS (IN-#)				EQUA. 12 $\frac{S_{LP} + \frac{(M_A + M_{VA})}{Z_p}}{1 - \frac{M_A + M_{VA}}{Z_p}}$	REMARK	
				T <sub>x</sub>	M <sub>y</sub>	M <sub>z</sub>	SRSS <sup>(1)</sup>	M <sub>x</sub>	M <sub>y</sub>	M <sub>z</sub>	SRSS <sup>(2)</sup>			
I	3" SFE. 112Z SD 166	4954	1222	8475	11540	5032	14695	16430	25792	21460	37350	40139	19375	< 0.700000. SAF.FACTOR
II	6" SEP 40 SD ELL	2957	626	+37	959	525	2286	26266	4359	278.10	38374	38442	7553	
12		2957	1244	1586	1746	1432	2760	17300	30300	20550	40000	40093	7820	
14		2957	1367	1586	1127	25.87	3705	14300	32910	21160	41866	42030	8062	
15		2957	2524	571	1493	1258	2034	15860	21560	1357	31826	31891	7007	
20		2957	4129	826	1507	1587	2339	1251	14210	33720	37532	37606	7867	
24	6" SEP 40 TEE SD 40 1/2-Z	2957	2273	1035	1813	2985	3169	15350	65080	54220	80086	80144	23850	
72	6" SEP 40 PIPE	2957	5649	107	1194	1910	2364	16330	30780	49790	60942	61916	10787	
25	3" SFE. 112 SD 166	2957	1204	2034	2557	4083	5714	36510	34400	22350	34732	53030	24574	
32	6" SEP 40 ELL	2957	1544	1064	9.4	1655	2173	15810	13420	14910	2554	25633	8400	
26	6" SEP 40 SD ELL	2957	1185	675	1107	1749	2177	20590	23700	40270	51061	51108	9109	
31	"	2957	1825	1046	1619	1176	2405	16910	41110	28140	53296	53341	9455	

WALL :  $S_{LP} = \frac{P D}{4 t}$ ,  $P = 500 \text{ PSI}$

$M_B$  - RESULTANT MOMENTS DUE TO  
OBE & VALVE ACTUATION

i - STRESS INT. FACTOR

$Z_p$  - PIPE SECTION MODULUS

ENGINEERING DEPARTMENT  
CALCULATION SHEET

SHEET NO. 3

SUBJECT: S.P. VALVE SELECTION PIPING (SONGS-1)

SESSION CALCULATION NO. DC

DATE 5/20/85 BY H.C. DATE 5/21/85

I.D. NO.

7764

MATERIAL

A. ESCARIO

DATE

5/20/85

LINE NO.	TYPE OF COMPONENT	S.F.	$M_A$ (WEIGHT) IN-LB.	OEE MOMENTS (IN-#)				VALVE ACTUATION MOMENTS (IN-#)				$M_B$ $\frac{M_A}{(I_F \times k)}$	EQUA. 12 $C_{L4} + \frac{V_A \times I_{10}}{3L}$	REMARK
				$T_x$	$M_y$	$M_z$	SRSS <sup>(1)</sup>	$M_x$	$M_y$	$M_z$	SRSS <sup>(2)</sup>			
52	6" GATE AD S.S. PIPE	2957	1465	1061	1601	1085	2203	18310	38270	22240	50530	50570	9579	L 5049487 S.S. 50570
46	"	2957	4451	-12	17.92	1063	2203	10620	17550	58460	61671	61712	10740	/
73	6" GATE AD S.S.	2957	6515	A.02	976	225	132.4	31020	26.020	82280	97843	92424	14647	
43	6" BUTT TEE S.S. A.02, 504	2957	2685	14.7	3101	1221	2153	18500	20010	55330	78220	74312	21037	
61	10" GATE AD S.S. PIPE	3681	5181	534	3200	4131	64.43	40400	183400	78161	24734	24732	11970	
43	"	3681	3817	2961	4315	3243	6162	139700	148400	74980	217165	21753	11074	
71	10" FLANGE S.S. A.02, 3681	3681	7842	5165	1063	7778	9375	66050	207610	147610	263146	262214	12748	/

WATER :  $C_{L4} = \frac{V_L}{4L}$

$M_A$  - RESULTANT MOMENT DUE TO WEIGHT

$M_B$  - RESULTANT MOMENT DUE TO OEE & VALVE ACTUATION

$k$  - STRESS INT. FACTOR

$I_p$  - PIPE SECTION MODULES

PSI

ALLOWABLE ST. LCCS =  $1.8 C_h = 21000$  PSI  
FOR A-106 GR. B

**CALCULATION SHEET**

ENGINEERING DEPARTMENT

SUBJECT: S/S 1/1/15 DRAFTING DESIGN - 1 DESIGN CALCULATION NO. DC

JO NO.

MATERIAL

DATE 5/28/15

CHN. NO. 10541

DATE 5/23/15

Table Occasional Load Evaluation for Upstream Piping

NODEL PT	TYPE OF COMPONENTS	PRES. STRESS $\sigma_{sp} = \frac{PD_o}{4t}$	DN STRESS $\sigma_N = \frac{M_A}{Z_p}$	DYNAMIC LOAD (IN-LBS)			COMBINED STRESS $\sigma_{sp} + .75i \left( \sigma_N + \frac{M_B}{Z_p} \right)$	P.E.H.L.L.S
				SSE (MR <sub>1</sub> )	VALVE ACTUATION (MR <sub>2</sub> )	MB $\sqrt{M_{R_1}^2 + M_{R_2}^2}$		
1	3" PRES. NC3 SCH 160	4994	642	32510	31350	49516	22852	< 2.4 < 1.0 40224 ∴ SATISFACTION
2	3" L.R. ELL SCH 160	4994	454	29147	30925	42495	20223	"
3	3" L.R. ELL SCH 160	4994	347	24070	22196	32741	16725	"
4	3" FLG. JT.			7696	14578	16484		"
5	VALVE BODY			12564	37081	39151		"
6	6" PIPE SCH 40	2957	98	1037	21645	21665	5604	
25	3" PRES. NC3 SCH 160	4994	2642	42224	54732	69126	31672	
26	3" L.R. ELL SCH 160	4994	2443	36212	50211	61906	28962	"
27	3" L.R. ELL SCH 160	4994	2253	29193	41128	53047	25721	"
28	3" FLG. JT.			10086	35705	37102		"
29	VALVE BODY			20306	36836	42062		"
30	6" PIPE SCH 40	2957	476	16071	33672	37310	7821	< 2.4 $\sigma_b = 36000$ ∴ SATISFACTION

NOTE :  $.75i \leq 1.0$

$\sigma_b = 16760 \text{ psi } @ 650^\circ\text{F}$  (ASME TP316)

NRC Question No. 20

In the stress analysis of the piping system, only the stresses due to primary loads were evaluated. The secondary stresses, particularly due to thermal expansion, should also be considered in accordance with Equations 13 and 14 of the ANSI B31.1 Code. Provide an evaluation of these secondary stresses.

SCE Response

The secondary stresses due to thermal expansion were calculated for the San Onofre Unit 1 pressurizer safety/relief valve discharge piping in the return to service (RTS) task of 1984. A summary of thermal stresses for the most severe loading condition is attached. The maximum stress is 31,742 psi at the junction of RV 533 piping and common header. It is caused by the 1.93° upward movement of the pressurizer nozzle when upstream piping of the safety valve is at 640°F and downstream 100°F.

ENGINEERING DEPARTMENT  
CALCULATION SHEET

SUBJECT S/R. VALVE DISCH. PIPING (SONGS-1)

I.O. NO.

DESIGN DATE 5/13/85 CALCULATION NO. DC  
MADE BY A. ESCARIO

CHECKED BY C. M. CHANAYA 5/13/85

## SECONDARY STRESSES EVALUATION

DATA PT.	COMPONENT TYPE & MTL.	OUTSIDE DIAMETER	MATERIAL	THICK.	DESIGN PRES.	DESIGN TEMP. CABR-1 CABR-2	PIPE STRESSES W/ SIP APPLIED			STRESS RATIO		REMARKS
							LONG. PRES. STRESS ( $S_L$ )	WEIGHT STRESS ( $S_W$ )	THR. BIP. STRESS ( $S_B$ )	$\frac{S_e}{S_A}$	$\frac{S_L + S_W + S_B}{S_A + S_h}$	
							D <sub>o</sub>	t <sub>n</sub>	P			
1	3° PRES. NO2. A-312 TP-316	3.5	.437	2500	640	640	5005	425	13633 13068	0.49	0.43	PASSED
2	3° SCH. 160 ELL A-312 TP-316	3.5	.437	2500	640	640	5005	268	13761 12585	0.50	0.43	"
22	6° 5D BEND SCH 40 A-106 GR. B	6.625	.28	500	470	100	2957	650	13673 3876	0.61	0.46	"
24	10° RTEE SCH 40 A-106 GR. B	10.75	.365	500	470	100	3681	526	18795 4833	0.84	0.61	"
25	3° PRES. NO2. A-312 TP-316	3.5	.437	2500	640	640	5005	446	12199 17483	0.63	0.52	"
26	3° SCH 160 ELL A-312 TP-316	3.5	.437	2500	640	640	5005	386	11773 17591	0.64	0.52	"
32	6° SCH 40 ELL A-106 GR. B	6.625	.28	500	470	100	2957	310	7454 11479	0.51	0.39	"
49	10° RTEE SCH 40 A-312 TP-316	10.75	.365	500	470	100	3681	617	26717 4449	0.96	0.70	"

WHERE:  $S_A = f(1.25S_e + .25S_h)$

$$S_W = \frac{.75 i M_A}{z}$$

$$f = 1.0$$

$S_e$  - MTL ALLOW. @ COLD TEMP.

$$S_B = \frac{i M_c}{z}$$

$S_h$  - MTL ALLOW. @ HOT TEMP.

$$S_c = \frac{P D_o}{4 t_n}$$

CCS POWER PIPING CODE (ANSI/ASME B31.1 1980  
SECTION 104) FOR DEFINITION OF OTHER NOTATION.

	PIPE MATERIALS		
	A-312 TP 316	A-106 GR B	A-234 WP B
$S_e$	18000	15000	17500
$S_h$	1676 @ 640°F	16000 @ 470°F	17500 @ 470°F
$S_A$	21690	22500	26250
$S_A + S_h$	44450	37500	43750

ENGINEERING DEPARTMENT  
CALCULATION SHEET

SUBJECT: G/R VALVE DISCH. PIPING (SONGS-1)

ORIGIN CALCULATION NO. DC

J.O. NO.

A. ESCARIO

DATE 5/13/86

CMM BY

*E. Chaney 5/13/86*

SECONDARY STRESSES EVALUATION

DATA PT.	COMPONENT TYPE & MTL.	OUTSIDE DIAMETER D.	NOM. THICK. t <sub>n</sub>	DESIGN PRES. P	DESIGN TEMP. CARR-1 CARR-2	PIPE STRESSES W/ SIP APPLIED			STRESS RATIO		REMARKS
						LONG. PRES. STRESS (S <sub>L</sub> )	WEIGHT STRESS (S <sub>W</sub> )	THR. BIP. STRESS (S <sub>B</sub> )	$\frac{S_e}{S_A}$	$\frac{S_L + S_W + S_B}{S_A + S_B}$	
						100	100	100			
24	10" RTEL SCH-40 A-106 GRB	10.75	.365	500	470 100	3681	75	31742 6159	1.41	0.95	PASSED
60	10" SCH. 40 ELL A-106 GRB	10.75	.365	500	470 100	3681	285	14653 4659	0.65	0.50	"
75	SOCKET WELD A-312 TP 316	2.375	.343	2500	640 640	4327	1143	20499 24736	0.89	0.68	"
76	SOCKET WELD A-312 TP 316	2.375	.343	2500	640 640	4327	887	18322 23463	0.85	0.65	"
78	SOCKET WELD A-312 TP 316	2.375	.343	2500	640 640	4327	1894	27838 29923	1.08	0.813	"
80	SOCKET WELD A-312 TP 316	2.375	.343	2500	640 640	4327	692	20647 21937	0.79	0.606	"
83	SOCKET WELD A-312 TP 316	2.375	.343	2500	640 640	4327	3378	13795 14140	0.51	0.49	"
91	4" x 2" OMC. RED A-234 NPB	4.5	.237	500	470 100	2573	257	11224 16533	0.63	0.44	"

WEFPE:  $S_A = f(1.25S_e + .25S_h)$

$f = 1.0$

$S_e$  - MTL ALLOW @ COLD TEMP.

$S_h$  - MTL ALLOW. @ HOT TEMP.

$$S_L = \frac{P.D.}{4t_n}$$

$$S_w = \frac{.75 + M_c}{2}$$

$$S_B = \frac{L M_c}{2}$$

SEE POWER PIPING CODE (ANSI/ASME B31.1 1980  
SECTION 104) FOR DEFINITION OF OTHER NOTATION.

	PIPE MATERIALS		
	A-312 TP 316	A-106 GRB	A-234 NPB
$S_e$	18800	15000	17500
$S_h$	16760 @ 141°F	15000 @ 10°F	17500 @ 10°F
$S_A$	17620	22500	26250
$S_A + S_h$	44450	37500	43750

L START J26 7:33:22 SCE 342 LCL 1  
L START J26 7:33:22 SCE 342 LCL 1

LX01 001 REMOTE12 CHEN  
LX01 001 REMOTE12 CHEN  
LX01 001 REMOTE12 CHEN  
LX01 001 REMOTE12 CHEN  
LX01 001 REMOTE12 CHEN

R00H E215 7:53:25 AM 23 MAY 34 012.PRI LX01 START 20  
R00H E215 7:53:25 AM 23 MAY 34 012.PRI LX01 START 20  
R00H E215 7:53:25 AM 23 MAY 34 212.PRI LX01 START 20  
R00H E215 7:53:25 AM 23 MAY 34 212.PRI LX01 START 20  
R00H E215 7:53:25 AM 23 MAY 34 212.PRI LX01 START 20

SONGS 1

Pressurizer

S/V Discharge

Piping

1. 1% Damping
2. with 2 snubbers removed

Valve-Actuation

Time-History  
Analysis

FOR THE ENGINEERING POD PLEASE CALL PAX 26646  
FOR DATA PROCESSING POD PLEASE CALL PAX 26075  
THE USER COMPUTING CENTER NUMBER IS PAX 26559

J E S 2 J O B L O G -- S Y S T E M A C 6 1 -- M O D E R S M U 2

01.21.33 JOB 7399 \$SHASP373 SCE34GLC STARTED - INIT 12 - CLASS C - SYS A781  
01.21.33 JOB 7399 JOBS SCE34GLC STARTED LMT=00:32:00 01.21.39 3 MAY 84  
01.21.33 JOB 7399 STEP PROGUSE SCE84GLC CPU 00:00:00 01.21.16 CC=0000  
01.21.33 JOB 7399 STEP INFO SCE84GLC CPU 00:00:00 01.21.11 CC=0000  
01.21.33 JOB 7399 \*IEF233A M 583 PRIVAT/SL/SCE84GLC,GC/PLOT.NONE  
01.21.33 JOB 7399 \$HASP375 SCE34GLC ESTIMATED LINES EXCEEDED  
01.32.12 JOB 7399 \$HASP375 SCE34GLC ESTIMATE EXCEEDED BY 10,000 LINES  
01.32.54 JOB 7899 \$HASP375 SCE84GLC ESTIMATE EXCEEDED BY 20,000 LINES  
01.31.28 JOB 7399 \$HASP375 SCE84GLC ESTIMATE EXCEEDED BY 30,000 LINES  
01.31.33 JOB 7399 IEF234E K 588/PVT/SCE84GLC,GC  
01.31.33 JOB 7399 STEP GC SCE84GLC CPU 00:07:45 01.31.39 CC=0000  
01.31.38 JOB 7399 JOB SCE24GLC ENDED CPU 00:07:45 01.31.39 3 MAY 84  
01.31.39 JOB 7399 &SHASP395 SCE84GLC ENDED

----- JES2 JOB STATISTICS -----

03 MAY 84 JOB EXECUTION DATE

345 CARDS READ

33,337 SYSOUT PRINT RECORDS

0 SYSOUT PUNCH RECORDS

3,635,881 SYSOUT SPOOL BYTES

10.50 MINUTES EXECUTION TIME

```

1 //SCE340LC J33 (75323769)C,E21S,CHEN,CLASS=C,TIME=30,REGION=2048K,    JCB 7399
2 // MSGLEVEL=1,MSGCLASS=A
3 *** EQ RSM02                                     SCE GENERATED STATEMENT
4 *** JDBPARM SYSAFF=(ACB1,5531)                 SCE GENERATED STATEMENT
5 *** LOGONID ECHMECH 1ACF1770 LOGONID SUCCESSFULLY SCANNED
6 *** PASS=FU 1ACF1770 PASS=GRD(S) SUCCESSFULLY SCANNED
7 *** PROUTE PRINT REMOTE12
8 //DD EXEC SCEP947,PGM=P767
9 XXSCEP947 PGM=PGM=P947,TIME=30,PRCLASS=4,PLOTNAME=None
10 ***
11 *** SAP V2 - PAP V2 - PIPING ANALYSIS PROGRAM
12 ***
13 XXPROGUSE EXEC PGM=PDSO,PARM=&GPGM
14 XXSTEPLI3 DD DSN=SCE.PROD.LOAD,DISP=SHR
15 XXFTX1FC01 DD DSN=SCE.PROGUSE.DATA,DISP=SHR
16 XXFTX6FC01 DD SYSOUT=&PRCLASS
17 ***
18 XXINFO EXEC PGM=IEBGGENER
19 XXSYSPRINT DD DUMMY
20 XXSYSIN DD DUMMY
21 XXSYSUT1 DD DSN=SCE.INFO.DATA,DISP=SHR
22 XXSYSUT2 DD SYSOUT=&PRCLASS,DCB=(RECFM=FA,BLKSIZE=80)
23 ***
24 XXGO EXEC PGM=&GPGM,TIME=&GTIME
25 XXSTEPLIB DD DSN=SCE.PROD.LOAD,DISP=SHR
26 XXFT01F001 DD UNIT=SYSDA,SPACE=(CYL,(20,5)),
27 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
28 XXFT02F001 DD UNIT=SYSDA,SPACE=(CYL,(50,5)),
29 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
30 XXFT03F001 DD UNIT=SYSDA,SPACE=(CYL,(20,5)),
31 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
32 XXFT04F001 DD UNIT=SYSDA,SPACE=(CYL,(20,5),RLSE),
33 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
34 XXFT05F001 DD DDNAME=SYSIN
35 XXFT06F001 DD SYSOUT=&PRCLASS
36 XXFT07F001 DD UNIT=SYSDA,SPACE=(CYL,(20,5)),
37 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
38 XXFT08F001 DD UNIT=SYSDA,SPACE=(CYL,(20,5)),
39 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
40 XXFT08F002 DD UNIT=SYSDA,SPACE=(CYL,(20,5)),
41 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
42 XXFT09F001 DD UNIT=SYSDA,SPACE=(CYL,(20,5)),
43 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
44 XXFT10F001 DD UNIT=SYSDA,SPACE=(CYL,(20,5)),
45 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
46 XXFT11F001 DD SYSOUT=B,
47 XX DCB=(RECFM=FB,LRECL=80,BLKSIZE=800)
48 XXFT12F001 DD UNIT=SYSDA,SPACE=(CYL,(20,5)),
49 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
50 XXFT13F001 DD UNIT=SYSDA,SPACE=(CYL,(20,5)),
51 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
52 XXFT14F001 DD UNIT=SYSDA,SPACE=(CYL,(20,5)),
53 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
54 XXFT15F001 DD UNIT=SYSDA,SPACE=(CYL,(20,5)),
55 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
56 XXFT21F001 DD UNIT=TAPED,DISP=(NEW,KEEP),DCB=DEN=3,LABEL=RETPD=5,
57 XX DSN=PLOT,&PLOTNAME
58 XXFT22F001 DD UNIT=SYSDA,SPACE=(CYL,(20,5)),
59 XX DCB=(RECFM=VBS,LRECL=1284,BLKSIZE=17980,BUFNO=2)
60 XXFT23F001 DD UNIT=SYSDA,SPACE=(CYL,(20,5)),

```

xx UCC=(RECFM=V05, LKEDCL=1284, CLASIZE=17935, UFND=2)  
\*\* LAST PROCEDURE UPDATE PERFORMED APRIL 26, 1934.  
//UC.SYSIN ED \*

JUJCS531  
09000540

## STMT NO. MESSAGE

```

6 IEF053I SUBSTITUTION JCL - PGM=PC50, PARM=P947
7 IEF053I SUBSTITUTION JCL - SYSCUT=A
12 IEF053I SUBSTITUTION JCL - SYSCUT=A, DCB=(RECFM=FA, BLKSIZE=30)
13 IEF053I SUBSTITUTION JCL - PGM=P947, TIME=10
21 IEF053I SUBSTITUTION JCL - SYSCUT=A
31 IEF053I SUBSTITUTION JCL - DSN=PLUT, NONE
IEF235I ALLOC. FOR SCE840LC PROGUSE GO
IEF237I 7AA ALLOCATED TO STEPLIB
IEF237I 7AA ALLOCATED TO SYS00004
IEF237I 7C4 ALLOCATED TO FTG1F001
IEF237I JES2 ALLOCATED TO FTG0F001
IEF142I SCE840LC PROCUSE GO - STEP WAS EXECUTED - COND CODE 0000
IEF295I SCE.PROD.LOAD KEPT
IEF285I VOL SER NOS= SCE003.
IEF285I STSCTLG.VSCE003. KEPT
IEF285I VOL SER NOS= SCE003.
IEF285I SCE.PROGUSE.DATA KEPT
IEF285I VOL SER NOS= SCE004.
IEF285I JES2.JOB07899.S00102 SYSOUT
IEF373I STEP /PROGUSE / START 86124.0121
IEF374I STEP /PROGUSE / STOP 86124.0121 CPU 0MIN 00.12SEC SRB 0MIN 00.01SEC VIRT 136K SYS 246K EXT CK SYS 3852K

```

## STEP END STATISTICS FOR STEP \*\*\* PROGUSE \*\*\* PROGRAM NAME \*\*\* POS0 \*\*\* 0000 COND CODE

01:21:09 STARTED	01:21:10 ENDED	00:00:01 ELAPSED	00:00:01 NET TIME	218 OPACITY
136K VIRT CORE USED	2048K VIRT REQD			105 DSP PRVY
				SEL PRVY

BASIC CHARGES- CPU TIME	=	.12 SECONDS	AT .05000 \$/SEC	*	.01
VOLUNTARY WAIT	=	.43 SECONDS	AT .00000 \$/SEC	*	:00
			BASIC SUB- TOTAL	*	:01*

DEVICE TYPE	UNIT ADDR	EXCP NUMBER	\$ EACH	NET SEC	\$/SEC
3380	07AA	5	.00020	.55	.00000
3380	07AA		.00020	:00	.00000
3380	07C4	3	.00020	.55	.00000

DEVICE CLASS	-----MOUNTING-----	---EXCP---	---OCCUP---			
	NUM SEA	CHARGE	CHARGE			
DASD	00	2.00	.00	.00	*	.00
TAPE	00	1.00	.00	.00	*	.00
U/R				.00	*	.00
OTHER			.00	.00	*	.00
			DEVICE SUB- TOTAL	*	.00*	

STEP SUB- TOTAL	.01 X 1.00 PRIORITY AND SHIFT FACTOR =	STEP TOTAL	*	\$ .01*
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IEF236I ALLOC. FOR SCE840LC INFO GO
IEF237I DMY ALLOCATED TO SYSPRINT
IEF237I DMY ALLOCATED TO SYSIN
IEF237I 7C4 ALLOCATED TO SYSUT1

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```

IEF237I 744 ALLOCATED TO SYSC0000
IEF237I JES2 ALLOCATED TO SYSUT2
IEF142I SCE340LC INFO GO - STEP WAS EXECUTED - COND CODE 0000
IEF235I SCE.INFO.DATA KEPT
IEF235I VOL SER NOSE= SCE004.
IEF235I SYSTLG.VSCE003. KEPT
IEF235I VOL SER NOSE= SCE003.
IEF235I JES2.JCBG739Y.300103. SYSOUT
IEF373I STEP /INFO / START 34124.0121
IEF374I STEP /INFO / STGP 34124.0121 CPU CMIN 00.07SEC SRB DMIN 00.00SEC VIRT 44K SYS 256K EXT 0K SYS 8452K

```

.....  
 STEP END STATISTICS FOR STEP \*\*\* INFO \*\*\* PROGRAM NAME \*\*\* IE8GENER \*\*\* 0000 COND CODE

01:21:11 STARTED 44K VIRT CORE USEC	01:21:11 ENDED 2048K VIRT REQD	00:00:01 ELAPSED	00:00:00 NET TIME	18.4 OPACITY 105 DSP PRTY SEL PRIT
--	-----------------------------------	------------------	-------------------	--

BASIC CHARGES- CPU TIME VOLUNTARY WAIT	=	:07 SECONOS :31 SECONOS	AT .35000 \$/SEC AT .00000 \$/SEC BASIC SUB- TOTAL	= * :00 = * :00 = * :00*
---	---	----------------------------	--	--------------------------------

DEVICE CHARGES-DEVICE TYPE	UNIT ADDR	-----EXCP----- NUMBER \$ EACH	-----OCCUP----- NET SEC \$/SEC
-------------------------------	--------------	----------------------------------	-----------------------------------

3380	07C4	2 :00020	:38 :00000
------	------	----------	------------

3380	07AA	:00020	:38 :00000C
------	------	--------	-------------

DEVICE CLASS	-----MOUNTING----- NUM SEA CHARGE	---EXCP--- CHARGE	--OCCUP-- CHARGE
-----------------	--------------------------------------	----------------------	---------------------

DASD TAPE U/R OTHER	00 2.00 :00	.00 .00 .00 .00	.00 .00 .00 .00	= * :00 = * :00 = * :00 = * :00*
------------------------------	-------------	--------------------------	--------------------------	---

DEVICE SUB- TOTAL				
-------------------	--	--	--	--

STEP SUB- TOTAL	.00 X 1.00 PRIORITY AND SHIFT FACTOR =	STEP TOTAL	=	\$0.00*
-----------------	--	------------	---	---------

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IEF236I ALLOC. FOR SCE840LC GO GO
IEF237I 7AA ALLOCATED TO STEPL18
IEF237I 7AA ALLOCATED TO SYS00008
IEF237I 220 ALLOCATED TO FT01F001
IEF237I A60 ALLOCATED TO FT02F001
IEF237I 12F ALLOCATED TO FT03F001
IEF237I 850 ALLOCATED TO FT04F001
IEF237I JES2 ALLOCATED TO FT05F001
IEF237I JES2 ALLOCATED TO FT06F001
IEF237I A51 ALLOCATED TO FT07F001
IEF237I 840 ALLOCATED TO FT08F001
IEF237I A51 ALLOCATED TO FT08F002
IEF237I 12F ALLOCATED TO FT09F001
IEF237I A51 ALLOCATED TO FT10F001
IEF237I JES2 ALLOCATED TO FT11F001
IEF237I A40 ALLOCATED TO FT12F001
IEF237I 843 ALLOCATED TO FT13F001
IEF237I 844 ALLOCATED TO FT14F001
IEF237I 220 ALLOCATED TO FT15F001

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IEF237I SCE ALLOCATED TO FT21F001  
 IEF237I A43 ALLOCATED TO FT22F001  
 IEF217I 853 ALLOCATED TO FT23F001  
 IEF142I SCE 340LC GO - STEP WAS EXECUTED - COND CODE 0000  
 IEF245I SCE .PROG.LOAD KEPT  
 IEF225I VOL SER NOS=SCE003.  
 IEF235I SYSCFG.VSCE003. KEPT  
 IEF265I SYS84124.T012109.RA000.SCE840LC.R0000001 DELETED  
 IEF235I VOL SER NOS=SCRAD05  
 IEF235I SYS84124.T012109.RA000.SCE840LC.R0000002 DELETED  
 IEF285I VOL SER NOS=SCRAD03  
 IEF285I SYS84124.T012109.RA000.SCE840LC.R0000003 DELETED  
 IEF285I VOL SER NOS=SCRAD00  
 IEF285I SYS84124.T012109.RA000.SCE840LC.R0000004 DELETED  
 IEF285I VOL SER NOS=SCRAD01  
 IEF285I JES2.JOB07899.SIC101  
 IEF285I JES2.JOB07899.S00104 SYSIN  
 IEF285I SYS84124.T012109.RA000.SCE840LC.R0000005 SYSOUT  
 IEF285I VOL SER NOS=SCRAD04 DELETED  
 IEF285I SYS84124.T012109.RA000.SCE840LC.R0000006 DELETED  
 IEF285I VOL SER NOS=SCRAD02  
 IEF285I SYS84124.T012109.RA000.SCE840LC.R0000007 DELETED  
 IEF285I VOL SER NOS=SCRAD04  
 IEF285I SYS84124.T012109.RA000.SCE840LC.R0000008 DELETED  
 IEF285I VOL SER NOS=SCRAD06  
 IEF285I SYS84124.T012109.RA000.SCE840LC.R0000009 DELETED  
 IEF285I VOL SER NOS=SCRAD04  
 IEF285I JES2.JOB07899.S00105 SYSOUT  
 IEF285I SYS84124.T012109.RA000.SCE840LC.R0000010 DELETED  
 IEF285I VOL SER NOS=SCRAD03  
 IEF285I SYS84124.T012109.RA000.SCE840LC.R0000011 DELETED  
 IEF285I VOL SER NOS=SCRAD02  
 IEF285I SYS84124.T012109.RA000.SCE840LC.R0000012 DELETED  
 IEF285I VOL SER NOS=SCRAD02  
 IEF285I SYS84124.T012109.RA000.SCE840LC.R0000013 DELETED  
 IEF285I VOL SER NOS=SCRAD05  
 IEF285I SYS84124.T012109.RA000.SCE840LC.R0000014 DELETED  
 IEF285I VOL SER NOS=SCRAD03  
 IEF285I SYS84124.T012109.RA000.SCE840LC.R0000015 DELETED  
 IEF285I VOL SER NOS=SCRAD01  
 IEF372I STEP /GO / START 64124.0131 CPU 7MIN 65.07SEC SR8 0MIN 06.18SEC VIRT 1152K SYS 364K EXT OK SYS 8852K

---

\* STEP END STATISTICS FOR STEP \*\*\* GO \*\*\* PROGRAM NAME \*\*\* P947 \*\*\* 0000 COND CODE  
 \* 01:21:12 STARTED 01:31:39 ENDED 00:10:27 ELAPSED 00:10:23 NET TIME 76.6 OPACITY  
 \* 1152K VIRT CORE USED 2048K VIRT REQD 105 DSP PRTY  
 \* SEL PRTY  
 \* BASIC CHARGES- CPU TIME = 665.07 SECONDS AT .05000 \$/SEC = 23.25  
 \* VOLUNTARY WAIT = 157.66 SECONDS AT 00000 \$/SEC = .00  
 \* BASIC SUB- TOTAL = 23.25\*  
 \* DEVICE CHARGES-DEVICE TYPE UNIT -----EXCP----- -----OCCUP-----  
 \* TYPE ADDR NUMBER \$ EACH NET SEC \$/SEC  
 \* 3330 07AA 91 .00020 622.71 .00000  
 \* 3380 07AA .00020 .00 .00000

33800	0220	17	.00020	.022.71	.00000
33800	0440	5,314	.00020	.022.71	.00000
33800	042F	75	.00020	.022.71	.00000
33800	0850	47	.00020	.022.71	.00000
33800	0451	3,870	.00020	.022.71	.00000
33800	0451	21	.00020	.022.71	.00000
33800	0451	0	.00020	.00000	.00000
33800	0451	1	.00020	.00000	.00000
33800	0840	1	.00020	.00000	.00000
33800	0840	1	.00020	.00000	.00000
33800	0840	1	.00020	.00000	.00000
33800	0840	1	.00020	.00000	.00000
33800	0840	1	.00020	.00000	.00000
33800	0850	0	.00020	.00000	.00000

DEVICE CLASS	MOUNTING			---EXCP---	--OCCUP---		
	NUM	SEA	CHARGE			CHARGE	CHARGE
GASD	00	2.00	.00	1.89	.00	=	1.89
TAPE	00	1.00	.00	.00	.00	=	.00
U/R				.00	.00	=	.00
OTHER				.00	.00	=	.00
				DEVICE SUB- TOTAL		=	1.89*

STEP SUB- TOTAL      25.14 X    1.00 PRIORITY AND SHIFT FACTOR =     STEP TOTAL      =    \$25.14\*

I EF375I JOB /SCE840LC/ START 84124.0121  
 I EF376I JOB /SCE840LC/ STOP 84124.0131 CPU    7MIN 45.26SEC SR8    0MIN 06.19SEC

JOB PERFORMANCE ANALYSIS AND CHARGES FOR USE OF SO CAL EDISON COMPANY      SYSID-A0 VS/2 RELEASE-03.8 MODEL- 84

JOBNAME - SCE840LC	ACCOUNT NO. - 7838876900	DATE - 3 MAY 84	JOB CLASS - C	76.6 OPACITY
01:21:09 STARTED	01:31:39 ENDED	00:10:30 ELAPSED	00:10:24 NET TIME	JOB PRTY JAS RELEASE 1.0.0

BASIC CHARGES- CPU TIME	=	465.26 SECONDS	AT .05000 \$/SEC	=	23.26
VOLUNTARY WAIT	=	158.38 SECONDS	AT .00000 \$/SEC	=	.00
			BASIC SUB- TOTAL	=	23.26*

DEVICE CLASS	MOUNTING			---EXCP---	--OCCUP---		
	NUM	SEA	CHARGE			CHARGE	CHARGE
DASD	00	2.00	.00	1.89	.00	=	1.89
TAPE	00	1.00	.00	.00	.00	=	.00
U/R				.00	.00	=	.00
OTHER				.00	.00	=	.00
				DEVICE SUB- TOTAL		=	1.89*

PRIORITY AND SHIFT TOTAL =      .00    JOB TOTAL =      25.15 JOB CHARGE      =    \$25.15\*

PPPPPPPPPPPPP	AAAAAA	PPPPPPPPPPPPP	VV	VV
PPPPPPPPPPPPP	AAAAAAA	PPPPPPPPPPPPP	VV	VV
PP	AA	PP	VV	VV
PP	AA	PP	VV	VV
PP	AA	PP	VV	VV
PPPPPPPPPPPPP	AA	PPPPPPPPPPPPP	VV	VV
PPPPPPPPPPPPP	AA	PPPPPPPPPPPPP	VV	VV
PP	AA	PP	VV	VV
PP	AA	PP	VVV	VVV
PP	AA	PP	VVV	VVV
PP	AA	PP	VV	VV
PP	AA	PP	VV	VV

USC VERSION TWO / OCTOBER 1977  
SCE VERSION PAP1 / JUNE 23 1981

05/03/84 01:21:15

SONGS ONE S/R VALVE DISCHARGE PIPING TIME HISTORY ANALYSIS

## CONTINUOUS INFORMATION

NUMBER OF NODAL POINTS = 73  
 NUMBER OF ELEMENT TYPES = 1  
 NUMBER OF LOAD CASES = 0  
 NUMBER OF FREQUENCIES = 1  
 ANALYSIS CODE (NDYN) = 4  
 1. STATIC  
 2. MODAL EXTRACTION  
 3. FORCED RESPONSE  
 4. RESPONSE SPECTRUM  
 5. DIRECT INTEGRATION  
 6. FREQUENCY RESPONSE  
 7. BUCKLING ANALYSIS  
 SOLUTION MODE (MODEX) = 0  
 EQ. 0, EXECUTION  
 EQ. 1, DATA CHECK  
 EQ.-1, EXECUTION & COMBINE BOUNDARY LOADS  
 NUMBER OF SUBSPACE  
 ITERATION VECTORS (NAD) = 0  
 EQUATIONS PER BLOCK = 00  
 TAPE10 SAVE FLAG (N10SV) = 00  
 GRAVITATIONAL CONSTANT = 386.4C  
 TOTAL BLANK COMMON (MTOT)=20000

BANDWIDTH MINIMIZATION IS REQUESTED

REQUIRED BLANK COMMON FOR THIS STEP= 877

## NODAL POINT INPUT DATA

NODE NUMBER	BOUNDARY CONDITION CODES			NODAL POINT COORDINATES		
	X	Y	Z	XX	YY	Z
1	0	0	1	0	0	1
2	0	0	0	0	0	0
3	0	0	0	0	0	0
4	0	0	0	0	0	0
5	0	0	0	0	0	0
6	0	0	0	0	0	0
7	0	0	0	0	0	0
8	0	0	0	0	0	0
9	0	0	0	0	0	0
10	0	0	0	0	0	0
11	0	0	0	0	0	0
12	0	0	0	0	0	0
13	0	0	0	0	0	0
14	0	0	0	0	0	0
15	0	0	0	0	0	0
16	0	0	0	0	0	0
17	0	0	0	0	0	0
18	0	0	0	0	0	0
19	0	0	0	0	0	0
20	0	0	0	0	0	0
21	0	0	0	0	0	0
22	0	0	0	0	0	0
23	0	0	0	0	0	0
24	0	0	0	0	0	0
25	0	0	0	0	0	0
26	0	0	0	0	0	0
27	0	0	0	0	0	0
28	0	0	0	0	0	0
29	0	0	0	0	0	0
30	0	0	0	0	0	0
31	0	0	0	0	0	0
32	0	0	0	0	0	0
33	0	0	0	0	0	0
34	0	0	0	0	0	0
35	0	0	0	0	0	0
36	0	0	0	0	0	0
37	0	0	0	0	0	0
38	0	0	0	0	0	0
39	0	0	0	0	0	0
40	0	0	0	0	0	0
41	0	0	0	0	0	0
42	0	0	0	0	0	0
43	0	0	0	0	0	0
44	0	0	0	0	0	0
45	0	0	0	0	0	0
46	0	0	0	0	0	0
47	0	0	0	0	0	0
48	0	0	0	0	0	0
49	0	0	0	0	0	0
50	0	0	0	0	0	0
51	0	0	0	0	0	0
52	0	0	0	0	0	0
53	0	0	0	0	0	0
54	0	0	0	0	0	0
55	0	0	0	0	0	0
56	0	0	0	0	0	0
57	0	0	0	0	0	0
58	0	0	0	0	0	0
59	0	0	0	0	0	0
60	0	0	0	0	0	0
61	0	0	0	0	0	0
62	0	0	0	0	0	0
63	0	0	0	0	0	0
64	0	0	0	0	0	0
65	0	0	0	0	0	0
66	0	0	0	0	0	0
67	0	0	0	0	0	0
68	0	0	0	0	0	0
69	0	0	0	0	0	0
70	0	0	0	0	0	0
71	0	0	0	0	0	0
72	0	0	0	0	0	0
73	0	0	0	0	0	0





72  
73  
71  
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-15.000  
-15.141  
-25.543  
-49.138  
-49.182  
-49.438  
-98.438  
-25.000

-15.141  
-25.543  
-49.138  
-49.182  
-49.438  
-98.438  
-112.500

-55.555  
-52.511  
192.559  
171.359  
145.510  
123.661  
64.500

470.000  
470.000  
470.000  
470.000  
470.000  
470.000  
470.000

## EQUATION NUMBERS

4	1	2	3	4	5	6
12	13	14	15	16	17	18
23	24	25	26	27	28	29
35	36	37	38	39	40	41
49	50	51	52	53	54	55
67	68	69	70	71	72	73
85	86	87	88	89	90	91
97	98	99	100	101	102	103
103	104	105	106	107	108	109
115	116	117	118	119	120	121
121	122	123	124	125	126	127
127	128	129	130	131	132	133
133	134	135	136	137	138	139
139	140	141	142	143	144	145
145	146	147	148	149	150	151
151	152	153	154	155	156	157
157	158	159	160	161	162	163
163	164	165	166	167	168	169
169	170	171	172	173	174	175
175	176	177	178	179	180	181
181	182	183	184	185	186	187
187	188	189	190	191	192	193
190	191	192	193	194	195	196
196	197	198	199	200	201	202
202	203	204	205	206	207	208
208	209	210	211	212	213	214
214	215	216	217	218	219	220
220	221	222	223	224	225	226
226	227	228	229	230	231	232
232	233	234	235	236	237	238
238	239	240	241	242	243	244
244	245	246	247	248	249	250
250	251	252	253	254	255	256
256	257	258	259	260	261	262
262	263	264	265	266	267	268
268	269	270	271	272	273	274
274	275	276	277	278	279	280
280	281	282	283	284	285	286
286	287	288	289	290	291	292
292	293	294	295	296	297	298
298	299	300	301	302	303	304
304	305	306	307	308	309	310
310	311	312	313	314	315	316
316	317	318	319			

5	5	5	5
61	62	63	64
65	66	67	68
69	70	71	72
73			
1	2	3	4
5	6	7	8
9	10	11	12
13	14	15	16
17	18	19	20
21	22	23	24
25	26	27	28
29	30	31	32
33	34	35	36
37	38	39	40
41	42	43	44
45	46	47	48
49	50	51	52
53	54	55	56
57	58	59	60
61	62	63	64
65	66	67	68
69	70	71	72
73			

PIPE ELEMENT INPUT DATA

CONTROL INFORMATION

REQUIRED BLANK COMMON FOR THIS STEP = 734  
NUMBER OF PIPE ELEMENTS = 72

NUMBER OF MATERIAL SETS = 1

MAXIMUM NUMBER OF MATERIAL  
TEMPERATURE INPUT POINTS = 1

NUMBER OF SECTION PROPERTY SETS = 3

NUMBER OF BRANCH POINT NODES = 0

MAXIMUM NUMBER OF TANGENTS  
COMMON TO A BRANCH POINT = 6

FLAG FOR NEGLECTING AXIAL  
DEFORMATIONS IN BEND ELEMENTS = 0  
(EQ.1, NEGLECT)

MATERIAL PROPERTY TABLES

MATERIAL NUMBER = ( 1 )

NUMBER OF

TEMPERATURE POINTS = ( 1 )

IDENTIFICATION = ( CARBON STEEL )

POINT NUMBER	TEMPERATURE	YOUNG'S MODULUS	POISSON'S RATIO	THERMAL EXPANSION
1	470.00	27900000.0	0.300	0.696D-05

## SECTION PROPERTY TABLE

SECTION NUMBER	OUTSIDE DIAMETER	WALL THICKNESS	SHAPE FACTOR FOR SHEAR	WEIGHT/ UNIT LENGTH	MASS/ UNIT LENGTH	DESCRIPTION
1	4.625	0.500	0.2	2.15810+01	2.42920-02	
2	1.750	0.500	0.2	0.33750+01	0.37340-02	
3	1.500	0.438	0.0	0.14420+01	0.17120-02	

## ELEMENT LOAD CASE MULTIPLIERS

	CASE A	CASE B	CASE C	CASE D
X-DIRECTION GRAVITY	0.0	0.0	0.0	0.0
Y-DIRECTION GRAVITY	0.0	0.0	0.0	0.0
Z-DIRECTION GRAVITY	0.0	0.0	0.0	0.0
THERMAL DISTORTION	0.0	0.0	0.0	0.0
PRESSURE DISTORTION	0.0	0.0	0.0	0.0

## PIPE ELEMENT INPUT DATA

ELEMENT NUMBER	ELEMENT TYPE	NODE -I	NODE -J	MATERIAL NUMBER	SECTION NUMBER	REFERENCE TEMPERATURE (BEND RADIUS)	INTERNAL PRESSURE (THIRD POINT)	DIRECTION (X1-ORDINATE)	DIRECTION (Y1-ORDINATE)	COSINES (Z1-ORDINATE)	SINES (Z1-ORDINATE)	NODE INCREMENT (ALL FRACTION)	INPUT TAG
1	TANGT BEND	2	3	1	3	( 70.00 70.00 4.500)	100.00 100.00 ( ) ( )	0.0 0.0 ( ) ( )	0.0 0.0 ( ) ( )	0.0 0.0 ( ) ( )	0.0 0.0 ( ) ( )	1 1 3.0000	II IC 2
3	TANGT	3	4	1	3	70.00	100.00	0.0	0.0	0.0	0.0	1	II 12
4	TANGT	4	5	1	3	70.00	100.00	0.0	0.0	0.0	0.0	1	II 12
5	TANGT	5	6	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 12
6	TANGT	6	7	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 11
7	BEND	7	3	1	1	( 30.000)	( ) ( )	0.0 ( ) ( )	0.0 ( ) ( )	-63.500 ( ) ( )	-63.500 ( ) ( )	3.0000	IC 11
8	TANGT	8	9	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 11
9	TANGT	9	10	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 10
10	TANGT	10	11	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 10
11	BEND	11	12	1	1	( 30.000)	( ) ( )	0.0 ( ) ( )	-112.500 ( ) ( )	-63.500 ( ) ( )	-63.500 ( ) ( )	3.0000	IC 11
12	TANGT	12	13	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 12
13	TANGT	13	14	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 12
14	BEND	14	15	1	1	( 30.000)	( ) ( )	130.812 ( ) ( )	-112.500 ( ) ( )	-63.500 ( ) ( )	-63.500 ( ) ( )	3.0000	IC 12
15	TANGT	15	16	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 9
16	TANGT	16	17	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 9
17	BEND	17	18	1	1	( 30.000)	( ) ( )	130.812 ( ) ( )	-112.500 ( ) ( )	-5.500 ( ) ( )	-5.500 ( ) ( )	3.0000	IC 12
18	TANGT	18	19	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 12
19	TANGT	19	20	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 12
20	BEND	20	21	1	1	( 30.000)	( ) ( )	60.813 ( ) ( )	-112.500 ( ) ( )	64.500 ( ) ( )	64.500 ( ) ( )	3.0000	IC 12
21	TANGT	21	22	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 283
22	BEND	22	23	1	1	( 30.000)	( ) ( )	-38.188 ( ) ( )	-112.500 ( ) ( )	64.500 ( ) ( )	64.500 ( ) ( )	3.0000	IC 12
23	TANGT	23	24	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 12
24	TANGT	25	26	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 6
25	BEND	26	27	1	3	( 70.00 70.00 4.500)	( ) ( )	69.750 ( ) ( )	-20.313 ( ) ( )	0.0 ( ) ( )	0.0 ( ) ( )	3.0000	IC 12
26	TANGT	27	28	1	3	70.00	100.00	0.0	0.0	0.0	0.0	1	II 12
27	TANGT	28	29	1	3	70.00	100.00	0.0	0.0	0.0	0.0	1	II 12
28	TANGT	29	30	1	3	70.00	100.00	0.0	0.0	0.0	0.0	1	II 12
29	TANGT	30	31	1	3	70.00	100.00	0.0	0.0	0.0	0.0	1	II 12
30	BEND	31	32	1	3	( 30.000)	( ) ( )	69.750 ( ) ( )	0.0 ( ) ( )	-63.500 ( ) ( )	-63.500 ( ) ( )	3.0000	IC 12
31	TANGT	32	33	1	1	70.00	100.00	0.0	0.0	0.0	0.0	1	II 12
32	BEND	33	34	1	1	( 26.000)	( ) ( )	11.875 ( ) ( )	0.0 ( ) ( )	-63.500 ( ) ( )	-63.500 ( ) ( )	3.0000	IC 11

ELEMENT ELEMENT INPUT DATA													
ELEMENT NUMBER	ELEMENT TYPE	NODE -I	NODE -J	MATERIAL NUMBER	SECTION NUMBER	REFERENCE TEMPERATURE (BEND RADIUS)	INTERNAL PRESSURE (THIRD POINT)	DIRECTIGEN COORDINATES (X3-ORDINATE)	DIRECTIGEN COORDINATES (Y3-ORDINATE)	DIRECTIGEN COORDINATES (Z3-ORDINATE)	INCREMENT (WALL FRACTION)	NODE	INPUT TAG
33	TANGT	34	35	1	1	70.00	100.00	0.0	0.0	0.0	1	II	12
34	TANGT	35	36	1	1	70.00	100.00	0.0	0.0	0.0	1	II	12
35	BEND	36	37	1	1	( 30.000)	( )	( 11.675)( -94.500)( -63.500)( 3.0000)				IC	
36	TANGT	37	38	1	1	70.00	100.00	0.0	0.0	0.0	1	II	12
37	TANGT	38	39	1	1	70.00	100.00	0.0	0.0	0.0	1	II	12
38	BEND	39	40	1	1	( 30.000)	( )	( 130.812)( -94.500)( -63.500)( 3.0000)				IC	
39	TANGT	40	41	1	1	70.00	100.00	0.0	0.0	0.0	1	II	12
40	TANGT	41	42	1	1	70.00	100.00	0.0	0.0	0.0	1	II	9
41	BEND	42	43	1	1	( 30.000)	( )	( 130.812)( -94.500)( -5.500)( 3.0000)				IC	
42	TANGT	43	44	1	1	70.00	100.00	0.0	0.0	0.0	1	II	12
43	TANGT	44	45	1	1	70.00	100.00	0.0	0.0	0.0	1	II	12
44	BEND	45	46	1	1	( 30.000)	( )	( 60.813)( -94.500)( 66.500)( 3.0000)				IC	
45	TANGT	46	73	1	1	70.00	100.00	0.0	0.0	0.0	1	II	148
46	BEND	47	48	1	1	( 30.000)	( )	( -38.188)( -94.500)( 66.500)( 3.0000)				IC	
47	TANGT	48	49	1	1	70.00	100.00	0.0	0.0	0.0	1	II	12
48	TANGT	49	50	1	2	70.00	100.00	0.0	0.0	0.0	1	II	12
49	TANGT	50	24	1	2	70.00	100.00	0.0	0.0	0.0	1	II	155
50	TANGT	24	51	1	2	70.00	100.00	0.0	0.0	0.0	1	II	156
51	TANGT	51	52	1	2	70.00	100.00	0.0	0.0	0.0	1	II	10
52	TANGT	52	53	1	2	70.00	100.00	0.0	0.0	0.0	1	II	10
53	BEND	53	54	1	2	( 50.000)	( )	( -69.188)( -154.500)( 33.500)( 3.0000)				IC	
54	TANGT	54	55	1	2	70.00	100.00	0.0	0.0	0.0	1	II	12
55	BEND	55	56	1	2	( 50.000)	( )	( -15.188)( -208.500)( 33.500)( 3.0000)				IC	
56	TANGT	56	57	1	2	70.00	100.00	0.0	0.0	0.0	1	II	11
57	TANGT	57	58	1	2	70.00	100.00	0.0	0.0	0.0	1	II	10
58	TANGT	58	59	1	2	70.00	100.00	0.0	0.0	0.0	1	II	IC
59	BEND	59	60	1	2	( 15.000)	( )	( -15.188)( -346.500)( 33.500)( 3.0000)					
60	TANGT	60	61	1	2	70.00	100.00	0.0	0.0	0.0	1	II	12
61	BEND	61	62	1	2	( 50.000)	( )	( -49.188)( -346.500)( 67.500)( 3.0000)				IC	
62	TANGT	62	63	1	2	70.00	100.00	0.0	0.0	0.0	1	II	12
63	BEND	63	64	1	2	( 50.000)	( )	( -49.188)( -346.500)( 192.500)( 3.0000)				IC	



PIPE ELEMENT INPUT DATA												
ELEMENT NUMBER	ELEMENT TYPE	NODE -I	NODE -J	MATERIAL NUMBER	SECTION NUMBER	REFERENCE TEMPERATURE (DEG C)	INTERNAL PRESSURE (THIRD POINT)	DIRECTIONAL (X) (X3-ORDINATE)	DIRECTIONAL (Y) (Y3-ORDINATE)	COSINES (Z) (Z3-ORDINATE)	INCREMENT (WALL FRACTION)	NODE INPUT TAG
55	TANGT	54	65	1	2	70.00	100.00	0.0	0.0	0.0	1	II
55	TANGT	65	55	1	2	70.00	100.00	0.0	0.0	0.0	1	IC
60	BEND	65	67	1	2	( 50.000)	( 100.00)	( -49.188)( -499.520)	( 192.500)	( 3.0000)		
67	TANGT	57	63	1	2	70.00	100.00	0.0	0.0	0.0	1	II
63	TANGT	68	59	1	2	70.00	100.00	0.0	0.0	0.0	1	II
69	BEND	69	70	1	2	( 50.000)	( 100.00)	( 98.434)( -499.500)	( 123.661)	( 3.0000)		
70	TANGT	70	71	1	2	70.00	100.00	0.0	0.0	0.0	1	II
71	TANGT	72	22	1	1	70.00	100.00	0.0	0.0	0.0	1	II
72	TANGT	73	47	1	1	70.00	100.00	0.0	0.0	0.0	1	II

51	229	255	250	257	253	259
52	229	252	254	251	253	241
53	217	227	226	229	230	231
54	217	215	216	217	213	219
55	133	133	135	138	133	133
56	112	122	121	124	125	125
57	122	122	123	123	125	125
58	122	122	123	123	125	125
59	122	122	123	123	125	125
60	122	122	123	123	125	125
61	122	122	123	123	125	125
62	122	122	123	123	125	125
63	122	122	123	123	125	125
64	103	115	110	116	107	103
65	103	103	105	106	95	90
66	17	22	23	24	25	27
67	15	22	23	24	26	27
68	11	61	62	63	55	56
69	9	69	60	51	52	42
70	7	37	36	35	40	40
71	73	32	30	29	29	280
72	50	32	30	28	212	213
73	38	32	30	211	278	

BANDWIDTH PRIOR TO RESEQUENCING = 283  
 BANDWIDTH AFTER RESEQUENCING = 18

#### BANDWIDTH MINIMIZATION TIME LOG

READ FILE AND SET UP MINIMIZATION = 0.03  
 PERFORM THE MINIMIZATION = 0.01  
 REWRITE THE DATA FILES = 0.35  
 TOTAL FOR MINIMIZATION = 0.40  
 REQUIRED BLANK COMMON FOR THIS STEP = 1640

NUMBER OF EQUATIONS

TOTAL NUMBER OF EQUATIONS = 394  
SAMPLED = 19  
NUMBER OF EQUATIONS IN A BLOCK = 394  
NUMBER OF BLOCKS = 1

## TOTAL LOADS (STATIC) OR MASSES (DYNAMIC)

NUMBER	LOAD CASE	X-AXIS FORCE	Y-AXIS FORCE	Z-AXIS FORCE	X-AXIS MOMENT	Y-AXIS MOMENT	Z-AXIS MOMENT
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ALL EQUATIONS HAVE STIFFNESS ATTACHED

STRUCTURE LOAD CASE	ELEMENT	LOAD	MULTIPLIERS
1	3.0	3.0	0.0 0.0

STEP 0 - STEP SOLUTION CONTROL INFORMATION

NUMBER OF TIME VARYING FUNCTIONS = 20  
SECOND MOTION INDICATOR = 0  
EQU. NONE  
STATUS READ ACCELERATION INPUT  
NUMBER OF ARRIVAL TIMES = 0  
EQU. ALL FUNCTIONS ARRIVE AT TIME ZERO  
NUMBER OF SOLUTION TIME STEPS = 15000  
OUTPUT (PRINT) INTERVAL = 10  
SOLUTION TIME INCREMENT = 0.10000-03  
MASS-PROPORTIONAL DAMPING COEFFICIENT (ALPHA) = 0.0  
STIFFNESS-PROPORTIONAL DAMPING COEFFICIENT (BETA) = 0.53000-04

STIFFNESS MATRIX PARAMETERS

MINIMUM NON-ZERO DIAGONAL ELEMENT = 1.0110+05  
MAXIMUM DIAGONAL ELEMENT = 1.6190+10  
MAXIMUM/MINIMUM = 1.6000+05  
AVERAGE DIAGONAL ELEMENT = 4.7660+08  
DENSITY OF THE MATRIX = 21.4 PCT.

REQUIRED BLANK COMMON FOR THIS STEP= 1227

REQUIRED BLANK COMMON FOR THIS STEP= 16199

## DYNAMIC LOAD INPUT

NODE NUMBER	DEGREE OF FREEDOM	FUNCTION REFERENCE	ARRIVAL TIME NUMBER	FUNCTION MULTIPLIER
7	3	1	1	0.100000*01
7	3	12	1	0.103700*01
11	2	1	1	0.100000*01
11	2	11	1	0.100000*01
14	1	1	1	0.100000*01
15	1	1	1	0.100000*01
17	1	1	1	0.100000*01
17	1	1	1	0.100000*01
20	1	1	1	0.100000*01
20	1	1	1	0.100000*01
20	1	1	1	0.100000*01
22	1	1	1	0.100000*01
22	1	1	1	0.100000*01
24	1	1	1	0.100000*01
31	1	1	1	0.100000*01
31	1	1	1	0.100000*01
33	1	1	1	0.100000*01
33	1	1	1	0.100000*01
36	1	1	1	0.100000*01
39	1	1	1	0.100000*01
39	1	1	1	0.100000*01
42	1	1	1	0.100000*01
42	1	1	1	0.100000*01
43	1	1	1	0.100000*01
45	1	1	1	0.100000*01
45	1	1	1	0.100000*01
47	1	1	1	0.100000*01
47	1	1	1	0.100000*01
49	1	1	1	0.100000*01
53	1	1	1	0.100000*01
53	1	1	1	0.100000*01
59	1	1	1	0.100000*01
61	1	1	1	0.100000*01
61	1	1	1	0.100000*01
63	1	1	1	0.100000*01
66	1	1	1	0.100000*01
68	1	1	1	0.906000*00
98	3	20	1	-0.423000*00

REQUIRED BLANK COMMON FOR THIS STEP= 15762

## ARRIVAL TIME VALUES

INPUT ORDER	ARRIVAL TIME VALUE
-------------	--------------------

## TIME FUNCTION DATA

TIME FUNCTION NUMBER = ( 1)

NUMBER OF POINTS = ( 5)  
 SCALE FACTOR = ( -0.10000+01)  
 DESCRIPTION = (FORCING FUNCTION F1 )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP# 221

1	0.3	0.0
2	3.40000-02	0.13050+04
3	0.80000-02	0.13050+04
4	0.20000-01	0.13050+04
5	0.21000-01	0.0

TIME FUNCTION NUMBER = ( 2)

NUMBER OF POINTS = ( 15)  
 SCALE FACTOR = ( -0.10000+01)  
 DESCRIPTION = (FORCING FUNCTION F2 )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP# 621

1	0.0	0.0
2	0.40000-02	0.0
3	0.60000-02	0.17230+04
4	0.80000-02	0.22970+04
5	0.94000-02	0.22630+04
6	0.10000-01	0.16310+04
7	0.12000-01	0.14100+04
8	0.14000-01	0.16700+04
9	0.16000-01	0.20880+04
10	0.20000-01	0.20880+04
11	0.22000-01	0.15660+04
12	0.24000-01	0.78300+03
13	0.26000-01	0.36500+03
14	0.28000-01	0.0
15	0.15000+01	0.0

TIME FUNCTION NUMBER = ( 3)

NUMBER OF POINTS = ( 14)  
 SCALE FACTOR = ( 0.10000+01)  
 DESCRIPTION = (FORCING FUNCTION F3 )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1	0.0	0.0
2	0.16000-02	0.0
3	0.10000-01	0.38730+03
4	0.12000-01	0.22970+04
5	0.14000-01	0.31260+04
6	0.16000-01	0.31320+04
7	0.18000-01	0.13570+04
8	0.20000-01	0.73100+03
9	0.22000-01	0.15950+03
10	0.24000-01	0.14090+03
11	0.26000-01	0.94000+03
12	0.28000-01	0.52000+03
13	0.30000-01	0.0
14	0.15000+01	0.0

TIME FUNCTION NUMBER = ( 6 )

NUMBER OF POINTS = ( 12 )  
SCALE FACTOR = ( 0.10000+01 )  
DESCRIPTION = ( FORCING FUNCTION F6 )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1	0.0	0.0
2	0.14400-01	0.0
3	0.16000-01	0.24010+04
4	0.17000-01	0.33930+04
5	0.18000-01	0.20880+04
6	0.20000-01	0.57400+03
7	0.21000-01	0.36500+03
8	0.24000-01	0.15700+03
9	0.26000-01	0.26100+03
10	0.28000-01	0.52200+03
11	0.30000-01	0.0
12	0.15000+01	0.0

TIME FUNCTION NUMBER = ( 5 )

NUMBER OF POINTS = ( 11 )  
SCALE FACTOR = ( 0.70700+00 )  
DESCRIPTION = ( FORCING FUNCTION F5 )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1	0.0	0.0
2	0.17000-01	0.0
3	0.18000-01	0.19310+04
4	0.20000-01	0.40190+04
5	0.20600-01	0.43320+04
6	0.22000-01	0.78100+03
7	0.24000-01	0.26100+03

5 0.26050-01 0.26100+03  
6 0.28000-01 0.31300+03  
10 0.30050-01 0.3  
11 0.15000+01 0.0

TIME FUNCTION NUMBER = ( 6)

NUMBER OF POINTS = ( 9)  
SCALE FACTOR = ( -0.10000+01)  
DESCRIPTION = (FORCING FUNCTION F<sub>0</sub>)

INPUT TIME FUNCTION  
ORDER VALUE VALUE

REQUIRED BLANK COMMON FOR THIS STEP= 621

1 0.0 0.0  
2 0.20600-01 0.0  
3 0.22000-01 0.37060+04  
4 0.24000-01 0.48020+04  
5 0.26000-01 0.55850+04  
6 0.28000-01 0.60030+04  
7 0.30000-01 0.73600+04  
8 0.40000-01 0.73600+04  
9 0.15000+01 0.0

TIME FUNCTION NUMBER = ( 7)

NUMBER OF POINTS = ( 7)  
SCALE FACTOR = ( -0.70700+00)  
DESCRIPTION = (FORCING FUNCTION F<sub>M</sub>)

INPUT TIME FUNCTION  
ORDER VALUE VALUE

REQUIRED BLANK COMMON FOR THIS STEP= 621

1 0.0 0.0  
2 0.23700-01 0.0  
3 0.25700-01 0.46000+03  
4 0.39200-01 0.46000+03  
5 0.39300-01 0.0  
6 0.10000+00 0.0  
7 0.15000+01 0.0

TIME FUNCTION NUMBER = ( 8)

NUMBER OF POINTS = ( 7)  
SCALE FACTOR = ( -0.10000+01)  
DESCRIPTION = (FORCING FUNCTION F<sub>G</sub>)

INPUT TIME FUNCTION  
ORDER VALUE VALUE

REQUIRED BLANK COMMON FOR THIS STEP= 621

1 0.0 0.0

2	0.39100-01	0.0
3	0.39200-01	0.46000+03
4	0.74200-01	0.46000+03
5	0.74300-01	0.0
6	0.10900+00	0.0
7	0.15000+01	0.0

TIME FUNCTION NUMBER = ( 9)

NUMBER OF POINTS = ( 7)  
SCALE FACTOR = ( 0.73700+00)  
DESCRIPTION = (FORCING FUNCTION F E )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1	0.0	0.0
2	0.74100-01	0.0
3	0.74200-01	0.46000+03
4	0.10900+00	0.46000+03
5	0.10910+00	0.0
6	0.20000+00	0.0
7	0.15000+01	0.0

TIME FUNCTION NUMBER = ( 10)

NUMBER OF POINTS = ( 7)  
SCALE FACTOR = ( 0.10000+01)  
DESCRIPTION = (FORCING FUNCTION F D )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1	0.0	0.0
2	0.10890+00	0.0
3	0.10900+00	0.46000+03
4	0.12900+00	0.46000+03
5	0.12910+00	0.0
6	0.20000+00	0.0
7	0.15000+01	0.0

TIME FUNCTION NUMBER = ( 11)

NUMBER OF POINTS = ( 7)  
SCALE FACTOR = ( 0.10000+01)  
DESCRIPTION = (FORCING FUNCTION F C )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1	0.0	0.0
---	-----	-----

2 0.12390+00 0.0  
3 0.12980+00 0.46000+03  
4 0.17500+00 0.46000+03  
5 0.17510+00 0.0  
6 0.21500+00 0.0  
7 0.15000+01 0.0

TIME FUNCTION NUMBER = ( 12)

NUMBER OF POINTS = ( 7)  
SCALE FACTOR = ( -0.10000+01)  
DESCRIPTION = (FORCING FUNCTION F B )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1 0.0 0.0  
2 0.17490+00 0.0  
3 0.17500+00 0.46000+03  
4 0.21500+00 0.46000+03  
5 0.21510+00 0.0  
6 0.30000+00 0.0  
7 0.15000+01 0.0

TIME FUNCTION NUMBER = ( 13)

NUMBER OF POINTS = ( 7)  
SCALE FACTOR = ( -0.10000+01)  
DESCRIPTION = (FORCING FUNCTION F A )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1 0.0 0.0  
2 0.21490+00 0.0  
3 0.21500+00 0.46000+03  
4 0.23500+00 0.46000+03  
5 0.23510+00 0.0  
6 0.30000+00 0.0  
7 0.15000+01 0.0

TIME FUNCTION NUMBER = ( 14)

NUMBER OF POINTS = ( 7)  
SCALE FACTOR = ( -0.10000+01)  
DESCRIPTION = (FORCING FUNCTION F K )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1 0.0 0.0

2	0.23500-01	0.0
3	0.23700-01	0.0
4	0.23700-01	0.0
5	0.23800-01	0.0
6	0.10000-00	0.0
7	0.15000-01	0.0

TIME FUNCTION NUMBER = ( 15)

NUMBER OF POINTS = ( 7)  
 SCALE FACTOR = ( 0.70700+00)  
 DESCRIPTION = (FORCING FUNCTION F L )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1	0.0	0.0
2	0.25600-01	0.0
3	0.25700-01	0.90000+04
4	0.25800-01	0.90000+04
5	0.29400-01	0.0
6	0.10000+00	0.0
7	0.15000+01	0.0

TIME FUNCTION NUMBER = ( 16)

NUMBER OF POINTS = ( 7)  
 SCALE FACTOR = ( -0.10000+01)  
 DESCRIPTION = (FORCING FUNCTION F M )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1	0.0	0.0
2	0.29200-01	0.0
3	0.29300-01	0.90000+04
4	0.29400-01	0.90000+04
5	0.31700-01	0.0
6	0.10000+00	0.0
7	0.15000+01	0.0

TIME FUNCTION NUMBER = ( 17)

NUMBER OF POINTS = ( 7)  
 SCALE FACTOR = ( 0.70700+00)  
 DESCRIPTION = (FORCING FUNCTION F M )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1	0.0	0.0
---	-----	-----

1	0.31500-01	0.0
3	0.31850-01	x.96530+04
4	0.37500-01	0.0
5	0.37600-01	0.0
6	0.40000-01	0.0
7	0.15000+01	0.0

TIME FUNCTION NUMBER = ( 18 )

NUMBER OF POINTS = ( 7 )  
SCALE FACTOR = ( 0.10000+01 )  
DESCRIPTION = ( FORCING FUNCTION F P )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1	0.0	0.0
2	0.37400-01	0.3
3	0.37500-01	0.90000+04
4	0.46700-01	0.90000+04
5	0.46800-01	0.0
6	0.10000+00	0.0
7	0.15000+01	0.0

TIME FUNCTION NUMBER = ( 19 )

NUMBER OF POINTS = ( 7 )  
SCALE FACTOR = ( -0.10000+01 )  
DESCRIPTION = ( FORCING FUNCTION F Q )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1	0.0	0.0
2	0.44600-01	0.0
3	0.44700-01	0.90000+04
4	0.52400-01	0.90000+04
5	0.52500-01	0.0
6	0.10000+00	0.0
7	0.15000+01	0.0

TIME FUNCTION NUMBER = ( 20 )

NUMBER OF POINTS = ( 7 )  
SCALE FACTOR = ( 0.10000+01 )  
DESCRIPTION = ( FORCING FUNCTION F R )

INPUT ORDER	TIME VALUE	FUNCTION VALUE
-------------	------------	----------------

REQUIRED BLANK COMMON FOR THIS STEP= 621

1	0.0	0.0
---	-----	-----

2 0.52300-31 0.2000+34  
3 0.370-31 0.9000+34  
4 0.65100-31 0.9000+34  
5 0.55200-31 0.0000+00  
6 0.10000+31 0.0000+00  
7 0.15000+31 0.0000+00

REQUIRED BLANK COMMON FOR THIS STEP= 17164

DISPLACEMENT COMPONENT OUTPUT REQUESTS

CODE FOR OUTPUT TYPE = 1

E.1, HISTORY TABLE

E.2, PRINTER PLOT

E.3, MAXIMA ONLY

PRINTER PLOT SPACING = C

NODE DISPLACEMENT COMPONENT

NUMBER \* \* \* \* \*

5	1	2	3	0	0	0
11	1	2	3	0	0	0
14	1	2	3	0	0	0
29	1	2	3	0	0	0
36	1	2	3	0	0	0
39	1	2	3	0	0	0
0	0	0	0	0	0	0

## STRESS COMPONENT OUTPUT REQUESTS

CODE FOR OUTPUT TYPE = 1  
EG.1, HISTORY TABLE  
EG.2, PRINTER PLOT  
EG.3, MAXIMA ONLY  
PRINTER PLOT SPACING = 0

## ELEMENT TYPE (12)

ELEMENT NUMBER	*	*	*	*	*	*	*	*	*	*	*	*
1	1	2	3	4	5	6	7	8	9	10	11	12
8	7	8	9	10	11	12	13	14	15	16	17	18
12	1	2	3	4	5	6	7	8	9	10	11	12
13	1	2	3	4	5	6	7	8	9	10	11	12
14	10	11	12	13	14	15	16	17	18	19	20	21
15	7	8	9	10	11	12	13	14	15	16	17	18
16	7	8	9	10	11	12	13	14	15	16	17	18
19	7	8	9	10	11	12	13	14	15	16	17	18
21	7	8	9	10	11	12	13	14	15	16	17	18
23	7	8	9	10	11	12	13	14	15	16	17	18
24	7	8	9	10	11	12	13	14	15	16	17	18
30	7	8	9	10	11	12	13	14	15	16	17	18
32	7	8	9	10	11	12	13	14	15	16	17	18
33	1	2	3	4	5	6	7	8	9	10	11	12
34	1	2	3	4	5	6	7	8	9	10	11	12
35	1	2	3	4	5	6	7	8	9	10	11	12
36	1	2	3	4	5	6	7	8	9	10	11	12
37	1	2	3	4	5	6	7	8	9	10	11	12
40	1	2	3	4	5	6	7	8	9	10	11	12
43	7	8	9	10	11	12	13	14	15	16	17	18
45	7	8	9	10	11	12	13	14	15	16	17	18
51	7	8	9	10	11	12	13	14	15	16	17	18
52	7	8	9	10	11	12	13	14	15	16	17	18
56	7	8	9	10	11	12	13	14	15	16	17	18
57	7	8	9	10	11	12	13	14	15	16	17	18
58	7	8	9	10	11	12	13	14	15	16	17	18
62	7	8	9	10	11	12	13	14	15	16	17	18
66	7	8	9	10	11	12	13	14	15	16	17	18
65	7	8	9	10	11	12	13	14	15	16	17	18
70	7	8	9	10	11	12	13	14	15	16	17	18
71	7	8	9	10	11	12	13	14	15	16	17	18
72	7	8	9	10	11	12	13	14	15	16	17	18
73	0	0	0	0	0	0	0	0	0	0	0	0

REQUIRED BLANK COMMON FOR THIS STEP# 15286

EQUATION PARAMETERS

TOTAL NUMBER OF EQUATIONS = 394  
1/2 EQUATION BANDWIDTH = 18  
NUMBER OF EQUATIONS PER BLOCK = 394  
TOTAL NUMBER OF EQUATION BLOCKS = 1  
NUMBER OF COUPLING BLOCKS = 0

REQUIRED BLANK COMMON FOR THIS STEP= 15285

REQUIRED BLANK COMMON FOR THIS STEP= 10064













