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EXPERIMENTAL STUDY AND NUMERICAL MODELING OF NATURAL CONVECTION WITH CONDENSATION

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Outline

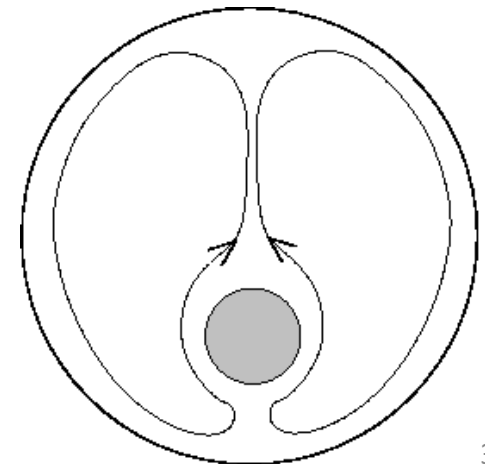
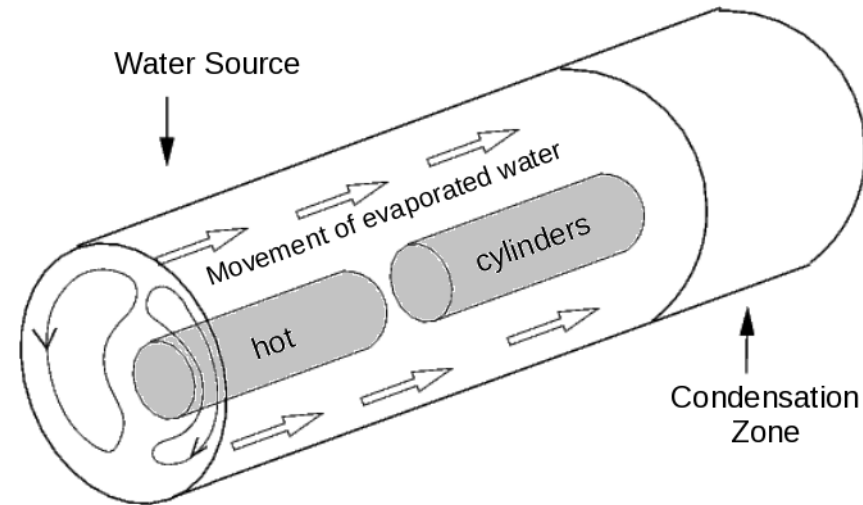


- ◆ Background
- ◆ Methodology
- ◆ Experimental Setup and Results
- ◆ Numerical Model Development
- ◆ Results of Numerical Study
- ◆ Comparison Between Experimental and Numerical Data
- ◆ Conclusions

Background: Relevant Processes



- ◆ Cold Trap Process
 - Seepage water
 - Radioactive decay heat
 - Evaporation on heated surface
 - Condensation on cold surfaces
 - Redistribution of water
- ◆ Physical Processes and Mechanisms
 - Natural convection
 - Driven by the thermal perturbation
 - Surface Condensation
 - Volumetric Condensation
 - Interphase heat transfer



Background: Objective of the Current Study



- ◆ Develop a Validated Numerical Tool
 - Capable of handling the physics
 - Eventual application to study cold trap process

- ◆ Technical Requirements of the Tool
 - Resolution of conjugate heat and mass transfer
 - Solution of species transport equations for water vapor
 - Simulation of surface condensation and evaporation
 - Simulation of volumetric condensation and latent heat transfer
 - Radiation heat transfer

- ◆ Validation Technique
 - Controlled experiment with single condensation-evaporation cycle

Methodology



- ◆ Numerical Simulations Using ANSYS® FLUENT® Version 12.1
- ◆ Customized Function Development for
 - Surface evaporation and condensation
 - Volumetric condensation
 - Interphase mass transfer between the liquid and vapor phase
- ◆ Experimental Study
 - Single source of water for evaporation
 - Measured condensation rate and temperature
- ◆ Comparison of Experimental Observation With Computed Data

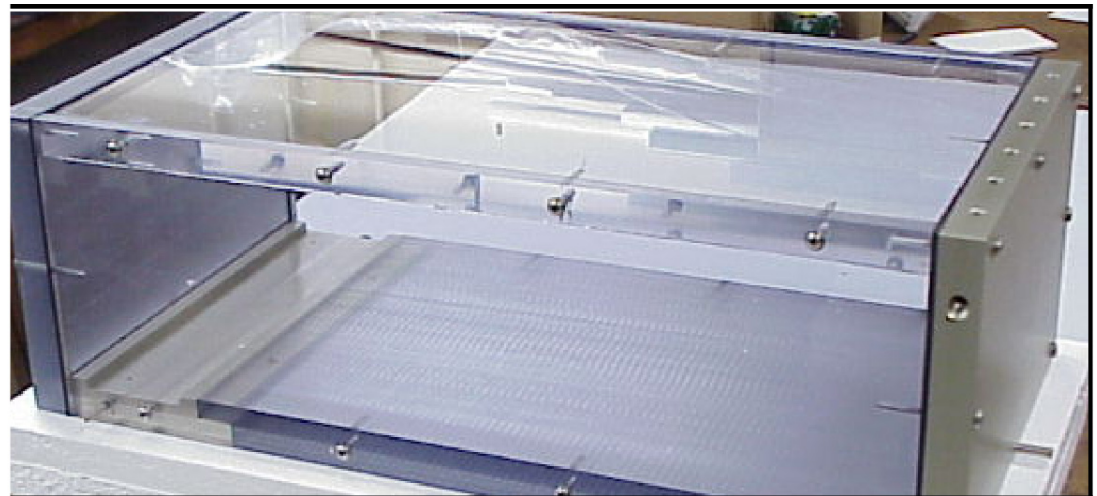
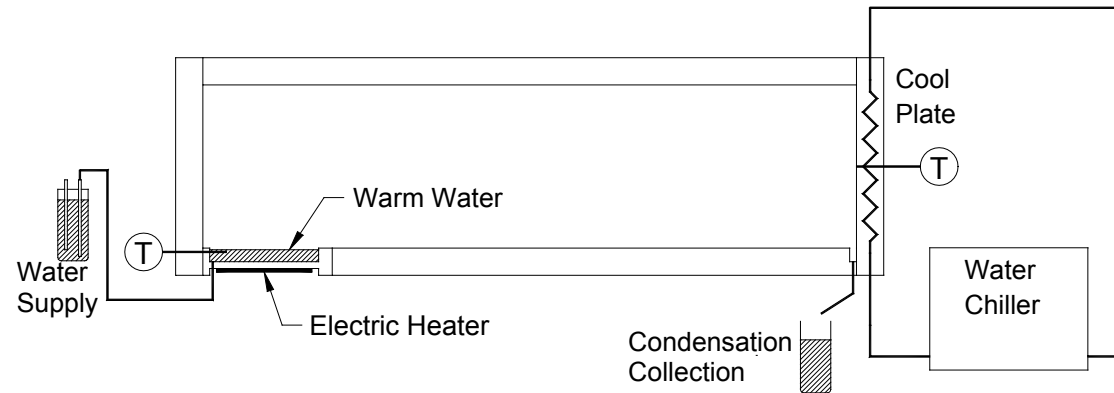
Experimental Setup



- ◆ Overall Enclosure Dimension:
23 × 6 × 12 Inches
- ◆ Materials: Polycarbonate Sheet and Acrylic Sheet
- ◆ Heated Tray for Evaporating Water With Continuous Supply
- ◆ Condensate Flow Collected in Graduated Cylinder
- ◆ Water Source Dimension
3 × 12 Inches
- ◆ Whole Chamber Covered With Polystyrene to Reduce Heat Loss
- ◆ Cooling Wall Material: Aluminum

1 in = [0.0254 m]

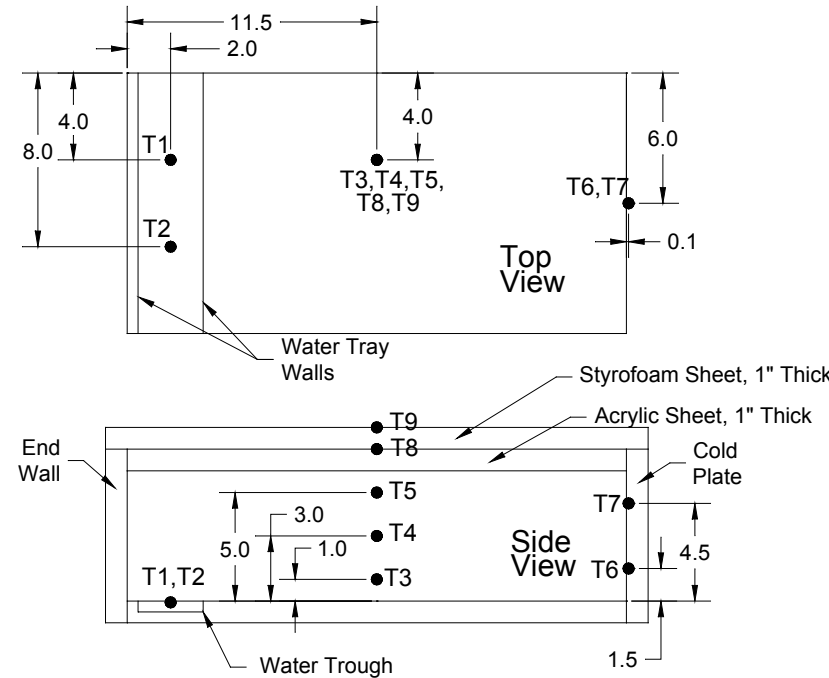
Schematic of the Setup



Experimental Setup: Instrumentation



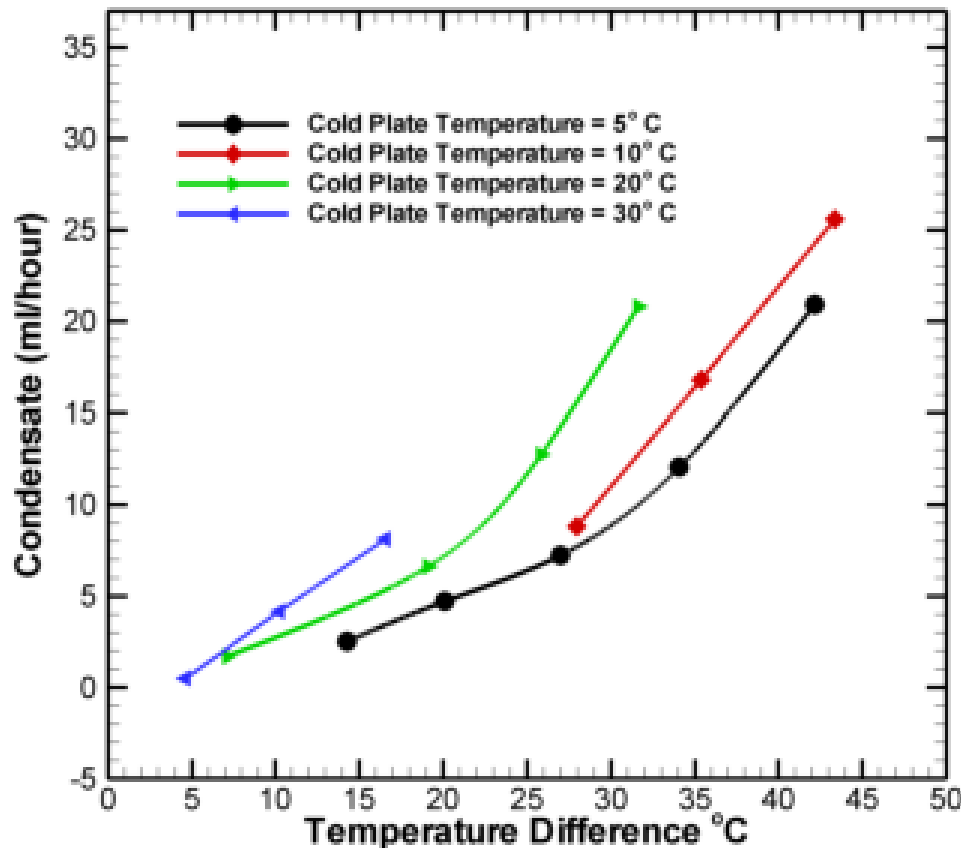
- ◆ Heater Pad for Water Source
 - Powered by variable AC supply
- ◆ Cooling Wall
 - Drilled channels with circulating cooling fluid
- ◆ Thermocouple Locations
 - Air temperatures at three locations
 - Evaporator temperature
 - Condenser temperature



all units are in inches

1 in = [0.0254 m]

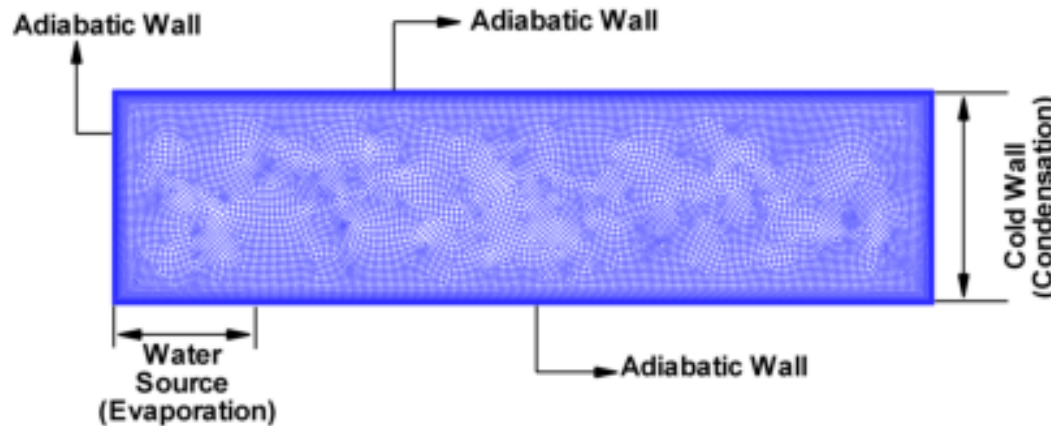
Experimental Results



- ◆ Experiment Conducted for Four Sets of Cold Plate Temperatures
- ◆ Temperature Difference = (Evaporator Temperature – Condenser Temperature)
- ◆ Condensation Rate Increases With Increased Temperature Difference

Temperature Difference: F = 1.8 C
 Temperature: F = 1.8 C + 32
 1 ml/hr = 2.64 10⁻⁴ gal/hr

Numerical Model Development



- ◆ ANSYS FLUENT 12.1
- ◆ Two-Dimensional Model
- ◆ Density-Driven Natural Convection Flow
- ◆ Approximated as Incompressible Fluid
 - Boussinesq approximation for density

- ◆ Navier-Stokes Equations
 - Momentum, species, and energy
- ◆ SIMPLEC for Pressure Velocity Coupling
- ◆ Shear Stress Transport (SST) k - ω Turbulence Model

Numerical Model Development: Treatment of Mass Transfer Process



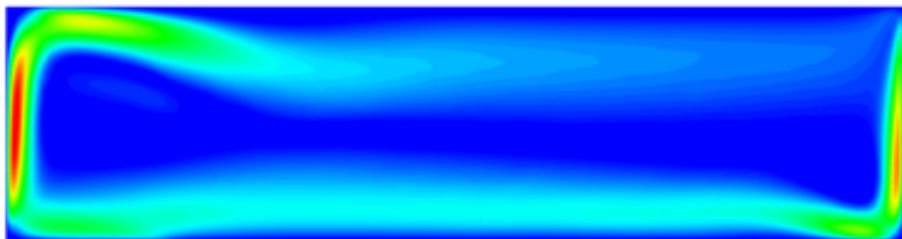
- ◆ Three Distinct Mass-Transfer-Related Processes
- ◆ Mass Transfer at the Walls Due to Condensation and Evaporation
 - Modeled using user-defined functions at the wall
 - Assumes equilibrium conditions adjacent to the wall
 - Only film condensation is considered
 - Latent heat exchange takes place from the source (i.e., wall)
 - Diffusion through mass transfer boundary layer
 - Mass, momentum, and energy source terms calculated based on condensation or evaporation rate and accounted for in the respective transport equations
 - Assumes equilibrium conditions adjacent to the wall
- ◆ Transport of Evaporated Water Vapor From the Hot Source Location
 - Solved the species transport equations using the mixture model

Numerical Model Development: Treatment of Mass Transfer Process (cont.)

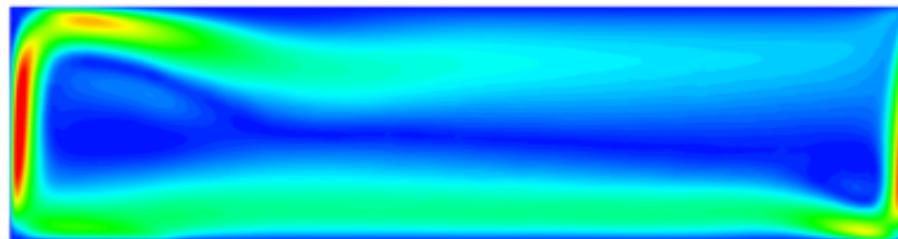


- ◆ Condensation of Water Vapor Within the Flow Domain: Two Different Modeling Approaches
 - Nonequilibrium model: Domain allowed to supersaturate (i.e., relative humidity can be more than 100%)
 - Supersaturation limit determined based on kinetic theory
 - Equilibrium Model: Volumetric Condensation Of Water Takes Place And Relative Humidity Restricted To 100%
 - Maximum mass transferred: Difference between the saturation and supersaturation limit
 - Condensed water flows with the air-vapor mixture
 - Rainout of droplets is not considered
 - Volumetric fraction of condensed liquid is significantly less compared to the main mixture

Results of Numerical Study: Velocity Field



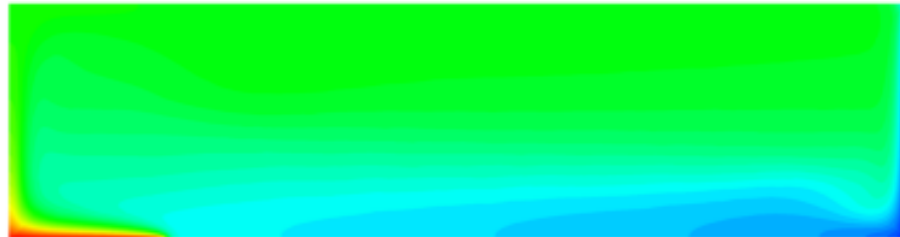
Nonequilibrium Model



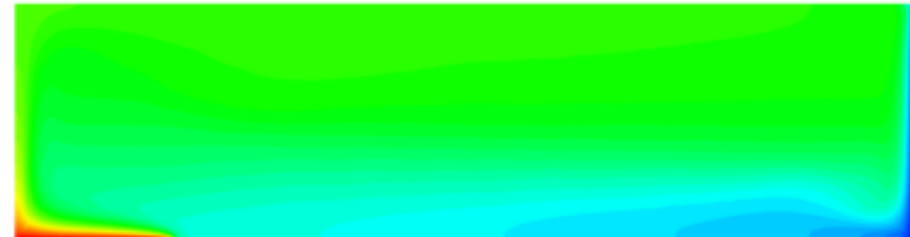
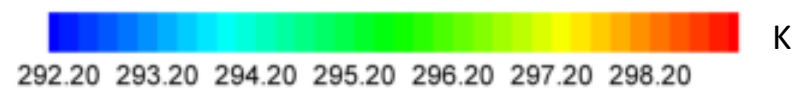
Equilibrium Model

1 m/s = 3.28 ft/s

Results: Temperature Field



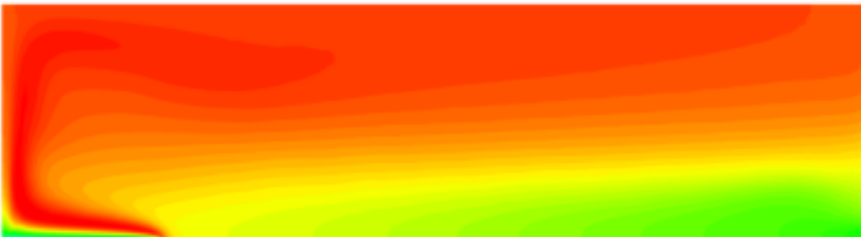
Nonequilibrium Model



Equilibrium Model

Temperature: F = 1.8 (K) -459.4

Results: Relative Humidity

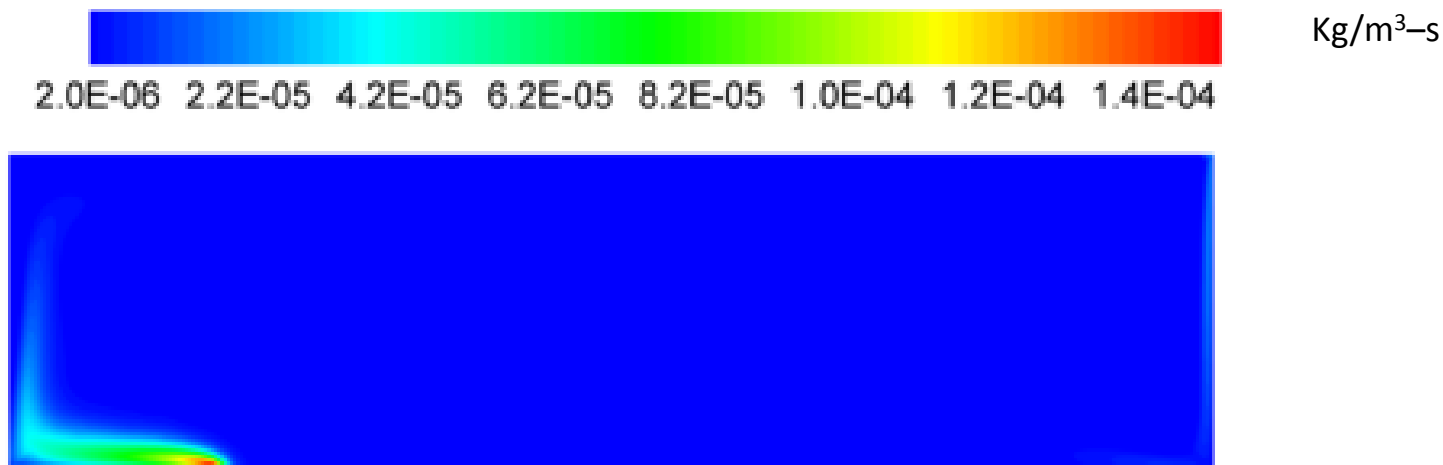


Nonequilibrium Model



Equilibrium Model

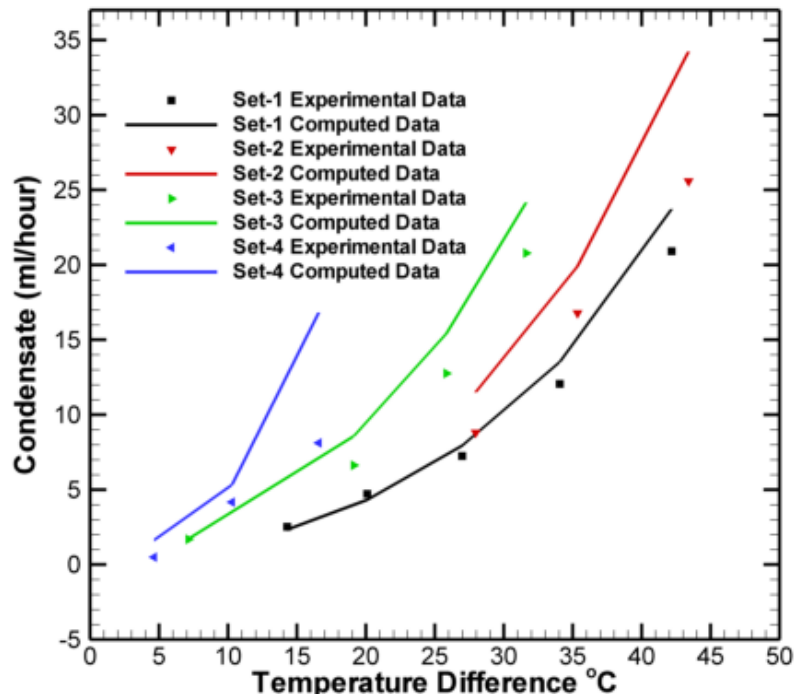
Results: Interphase Mass Transfer Rate



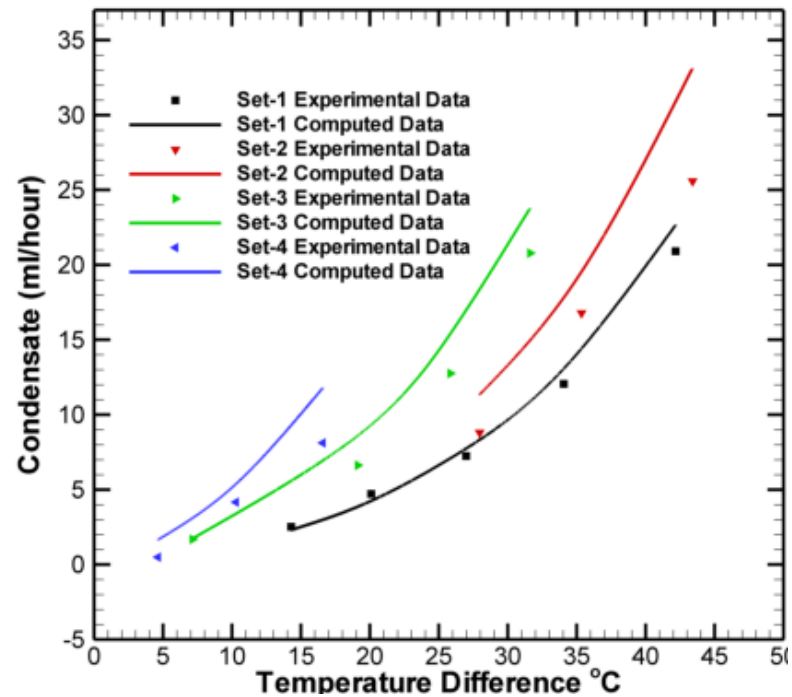
Equilibrium Model

$$1 \text{ Kg/m}^3\text{-s} = 6.242 \times 10^{-2} \text{ lb}_m/\text{ft}^3\text{-s}$$

Comparison Between Experimental and Numerical Data: Temperature



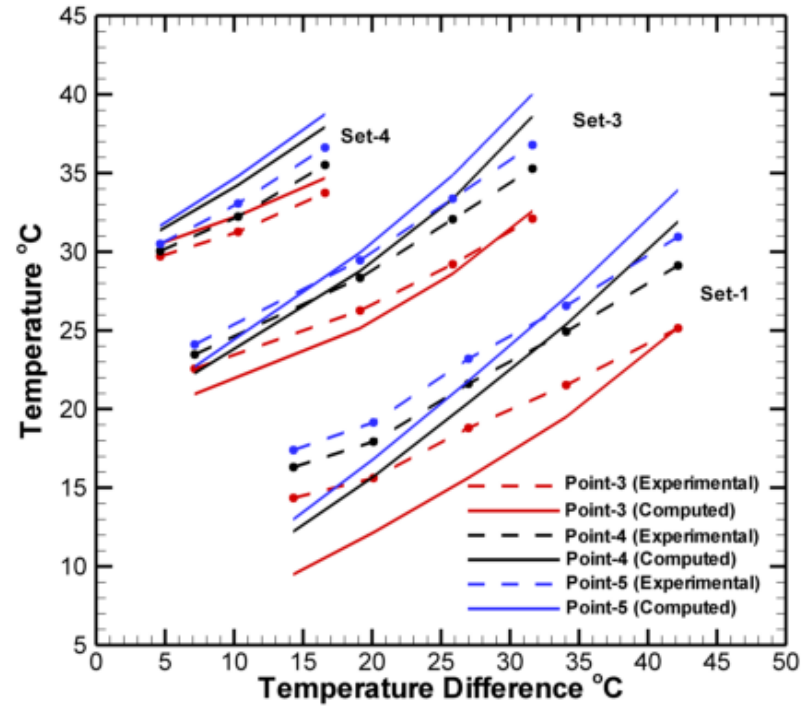
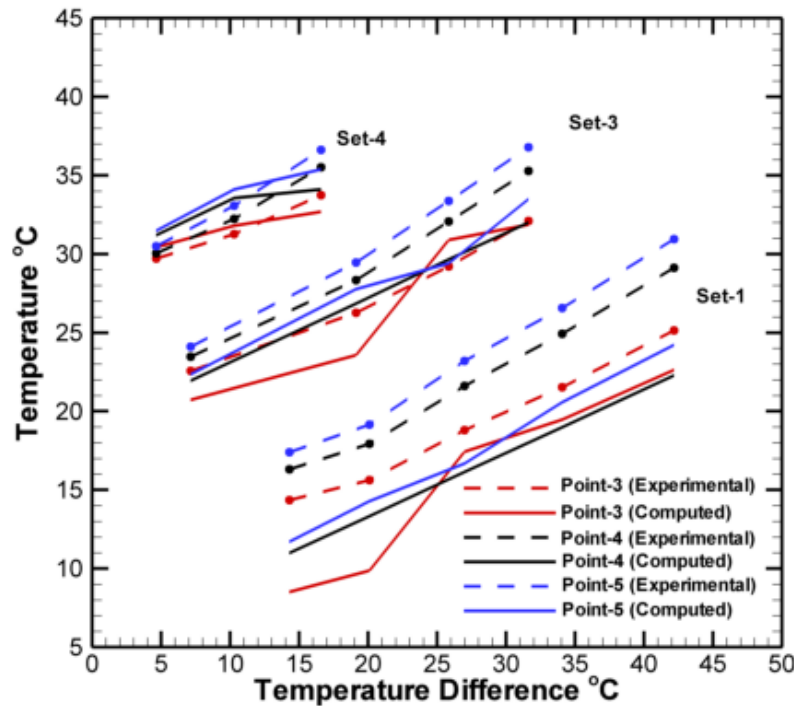
Nonequilibrium Model



Equilibrium Model

1 ml/hr = 2.64 $\times 10^{-4}$ gal/hr
Temperature Difference: F = 1.8 C

Comparison Between Experimental and Numerical Data: Condensation Rates



Temperature Difference: $F = 1.8 \quad C$
Temperature: $F = 1.8 \quad C + 32$

Conclusions

- ◆ A Combined Experimental and Numerical Study Was Conducted to Validate the Numerical Study
- ◆ Volumetric and Surface Condensation Modeling Tools Developed
- ◆ Both Equilibrium and Nonequilibrium Numerical Models for Volumetric Condensation Studied
- ◆ Equilibrium Model Provided Better Match With Experimental Data



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