

**ATTACHMENT (8)**

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**TRANSNUCLEAR CALCULATION NO. 1095-16 –  
NON-PROPRIETARY VERSION**

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Form 3.1-1  
Calculation Approval Sheet

Project Name: NUHOMS-32P Project #: 10950

Calculation Title: Transfer Thermal Analysis, -3°F Ambient

Calculation #: 1095-16 Draft/Revision #: 0 DCR #: ---

Number of pages: 20

Number of CDs attached: 1

If original issue, 10CFR72.48 review required?

No (explain)  Yes, SR No. \_\_\_\_\_

This calculation is performed in support of the licensee, CCNPP

1. This calculation is complete and ready for independent review

Originator's Signature *Dave V. Hoop* Date: 7/1/03

2. This calculation has been checked for consistency, completeness, and arithmetic correctness.

Checker Signature *[Signature]* Date: 7/1/03

3. Calculation preparation and check complies with procedure - package is complete

PE's Signature *Glenn Guena* Date: 8/25/03

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TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 1 OF 20  
CALC. NO 1095-16  
REV. 0

### 1.0 Purpose

To determine component temperatures of the NUHOMS-32P packaging during transport with an ambient temperature of -3 °F and no insolation using the same methodology as the Calvert Cliffs SAR supporting calculations.

### 2.0 References

- 1) Kreith et. al., The CRC Handbook of Thermal Engineering, 2000.
- 2) "The Topical Report for the NUTECH Horizontal Modular Storage System for Irradiated Nuclear Fuel", Rev. 1A.
- 3) Rohsenow et. al., Handbook of Heat Transfer Fundamentals, 1985.
- 4) Calculation 1095-5, Rev. 0, Finite Element Models, Thermal Analysis
- 5) (NOT USED)
- 6) *Chemical Engineers' Handbook, Fifth Edition*, Perry P.H. and Chilton C.H., McGraw-Hill Book Co., New York, 1973.
- 7) Calvert Cliffs Independent Spent Fuel Storage Installation, Safety Analysis Report, Volume 1, Rev. 11
- 8) ANSYS Computer Code and User's Manuals, Volumes 1-4, Rev. 5.6. See Test Reports E-17740, E-17741, and E-18554 through E-18557 for validation of computer code.
- 9) ANSYS files: /Calc1095-16/ CCCask.db, CCCask.rth, RadMat.sub, CCCask.mac  
CCBasket.db, CCBasket.rth, CCBasket.mac

### 3.0 Assumptions

This analysis is performed using the finite element models created within Reference 4. All assumptions, material properties, and modeling details are the same as those in Reference 4.

The material properties for air, used in the determination of the total heat transfer coefficients, are reprinted in Appendix A.

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 2 OF 20  
CALC. NO 1095-16  
REV. 0

#### 4.0 Discussion

The three-dimensional quarter-symmetry finite element models of the NUHOMS-32P packaging described in Reference 4 are used for the analysis. All assumptions and material properties from Reference 4 remain unchanged within this calculation. For conservatism, temperature distributions are determined under steady-state conditions.

Two finite element models are used within the analysis:

- A cask body model consisting of the cask body, lid, ram plate, and canister.
- A basket model consisting of the canister, basket, and fuel assemblies.

The cask body model is run with the fuel decay heat load applied as a heat flux to the canister inner cavity surfaces. The temperature distribution within the canister calculated within the cask body model is applied to the canister within the basket model. Noding of the canister within the two finite element models is identical. The fuel decay heat load is applied to the fuel elements of the basket model as an internal heat generation.

Boundary condition details can be found in the Appendices.

The approach used within the analysis is the same as that used in the supporting calculations for the Calvert Cliffs ISFSI SAR.

#### 4.1 Thermal Design Criteria

- A maximum fuel cladding temperature limit of 570 °C (1058 °F) is set for the fuel assemblies with an inert cover gas as concluded in Reference 7.
- A maximum temperature limit of 327 °C (620 °F) is set for the lead, corresponding to the melting point (Reference 6).
- A maximum limit of 280 °F on the bulk temperature of the resin is set in Reference 7.

**TRANSNUCLEAR, INC.**

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 3 OF 20  
 CALC. NO 1095-16  
 REV. 0

**5.0 Results**

Temperature distributions obtained from the finite element models are found in Appendix B.

**6.0 Conclusions**

Maximum Component Temperatures in NUHOMS-32P packaging

Component (---)	Maximum Temperature (°F)	Thermal Limits (°F)
Outer Shell	147	---
Resin	224	280 max.
Lead	265	620 max.
Cask Body	233	---
Fuel Cladding	664	1058 max.
Cask Lid	202	---
Ram Plate	132	---
Canister	371	---
Fuel Compartments	638	---
Peripheral Inserts	492	---
Aluminum Basket Plates	638	---
Stainless Steel Bars	638	---

All components remain below thermal design criteria during normal conditions of transport.

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 4 OF 20  
 CALC. NO 1095-16  
 REV 0

## APPENDIX A

### Total Heat Transfer Coefficients

The exposed portions of the packaging dissipate heat to the ambient via natural convection and radiation.

#### Radiation Coefficient

The heat transfer coefficient,  $H_r$ , for heat dissipation by radiation, is given by the equation:

$$H_r = F_{12} \left[ \frac{\sigma(T_1^4 - (E)(T_2^4))}{T_1 - T_2} \right] \text{ Btu/hr} - \text{ft}^2 - ^\circ\text{F}$$

where,

- $F_{12}$  = the gray body exchange coefficient
- = (surface emissivity) (view factor)
- $E$  = 1.0
- $T_1$  = surface temperature, °R
- $T_2$  = ambient temperature, °R

#### Emissivity and Absorptivity

The external surfaces of the packaging are stainless steel (emissivity = 0.587, Reference 2)

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 5 OF 20  
 CALC. NO 1095-16  
 REV. 0

Horizontal Cylinders

Heat dissipation by natural convection from horizontal cylinders is described by the following equations (Reference 1):

$$Nu = h_c \frac{L}{k} = \left( (Nu_l)^{15} + (Nu_t)^{15} \right)^{1/15}$$

$Nu_l$  = Laminar portion of the heat transfer coefficient.  
 $Nu_t$  = Turbulent portion of the heat transfer coefficient.

$$Nu_l = \overline{C}_l Ra^{1/3}$$

$$\overline{C}_l = 0.103 \text{ for } Pr \approx 0.71$$

$$Nu_l = \frac{2f}{\left( 1 + \frac{2f}{Nu^T} \right)}$$

$$f = 0.8 \text{ for } Ra > 10^{-2} ; Ra = Gr * Pr$$

$$Nu^T = 0.772 \left( \overline{C}_l Ra^{1/4} \right)$$

$$\overline{C}_l = \frac{0.671}{\left( 1 + \left( \frac{0.492}{Pr} \right)^{9/16} \right)^{4/9}}$$

$Nu^T$  accounts for the effect of the boundary layer thickness.  
 where,

- Nu = Nusselt number
- Gr = Grashof number =  $\rho^2 g \beta (T_s - T_a) L^3 / \mu^2$
- Pr = Prandtl number
- Ra = Rayleigh number = Gr \* Pr
- $\rho$  = density
- g = acceleration due to gravity
- $\beta$  = temperature coefficient of volume expansion
- $\mu$  = absolute viscosity
- L = characteristic length
- $h_c$  = natural convection coefficient
- k = thermal conductivity

The total heat transfer coefficients for radial surfaces of horizontal cylinders are calculated and input into ANSYS via the HTOT\_HCY macro, included within this appendix.

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 6 OF 20  
 CALC. NO 1095-16  
 REV. 0

Vertical Flat Plates

Heat dissipation by natural convection from vertical flat plates is described by the following equations (Reference 1):

$$Nu = h_c \frac{L}{k} = \left( (Nu_l)^6 + (Nu_t)^6 \right)^{1/6} \text{ for } 1 < Ra < 1012 ; Ra = Gr * Pr$$

$Nu_l$  = Laminar portion of the heat transfer coefficient.

$Nu_t$  = Turbulent portion of the heat transfer coefficient.

$$Nu_t = \frac{C_t^v Ra^{1/3}}{\left( 1 + 1.4 \times 10^9 \left( \frac{Pr}{Ra} \right) \right)}$$

$$C_t^v = \frac{0.13 Pr^{0.22}}{\left( 1 + 0.61 Pr^{0.81} \right)^{0.42}}$$

$$Nu_l = \frac{2.0}{\ln \left( 1 + \frac{2.0}{Nu^T} \right)}$$

$$Nu^T = \overline{C}_1 Ra^{1/4}$$

$$\overline{C}_1 = \frac{0.671}{\left( 1 + \left( \frac{0.492}{Pr} \right)^{9/16} \right)^{4/9}}$$

$Nu^T$  accounts for the effect of the boundary layer thickness.  
 where,

- Nu = Nusselt number
- Gr = Grashof number =  $\rho^2 g \beta (T_s - T_a) L^3 / \mu^2$
- Pr = Prandtl number
- Ra = Rayleigh number = Gr \* Pr
- $\rho$  = density
- g = acceleration due to gravity
- $\beta$  = temperature coefficient of volume expansion
- $\mu$  = absolute viscosity
- L = characteristic length
- $h_c$  = natural convection coefficient
- k = thermal conductivity

The total heat transfer coefficients for vertical flat surfaces are calculated and input into ANSYS via the HTOT\_VPL macro, included within this appendix.



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TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 7 OF 20  
 CALC. NO 1095-16  
 REV. 0

Total Heat Transfer Coefficients, Cask Radial Surfaces

Mat, 1

$\epsilon = 0.587$

View Factor = 1.0

L = 89.00"

$T_s$ (°F)	$T_{avg}$ (°F)	k (Btu/h-ft-°F)	Ra (--)	$H_c$ (Btu/h-in <sup>2</sup> -°F)	$H_r$ (Btu/h-in <sup>2</sup> -°F)	$H_{tot}$ (Btu/h-in <sup>2</sup> -°F)
110	53.5	0.0145	8.61E+10	0.0062	0.0038	0.0100
115	56.0	0.0146	8.79E+10	0.0062	0.0039	0.0101
120	58.5	0.0146	8.96E+10	0.0063	0.0040	0.0103
125	61.0	0.0147	9.12E+10	0.0064	0.0040	0.0104
130	63.5	0.0147	9.27E+10	0.0064	0.0041	0.0105
135	66.0	0.0148	9.41E+10	0.0065	0.0041	0.0106
140	68.5	0.0148	9.54E+10	0.0065	0.0042	0.0107
145	71.0	0.0149	9.66E+10	0.0066	0.0043	0.0109
150	73.5	0.0150	9.77E+10	0.0066	0.0043	0.0110
155	76.0	0.0150	9.87E+10	0.0067	0.0044	0.0111
160	78.5	0.0151	9.96E+10	0.0067	0.0045	0.0112
165	81.0	0.0152	1.00E+11	0.0068	0.0045	0.0113
170	83.5	0.0152	1.01E+11	0.0068	0.0046	0.0114
175	86.0	0.0153	1.02E+11	0.0069	0.0047	0.0116
180	88.5	0.0154	1.03E+11	0.0069	0.0047	0.0117
185	91.0	0.0154	1.03E+11	0.0070	0.0048	0.0118
190	93.5	0.0155	1.04E+11	0.0070	0.0049	0.0119
195	96.0	0.0155	1.04E+11	0.0071	0.0050	0.0120
200	98.5	0.0156	1.05E+11	0.0071	0.0050	0.0121
205	101.0	0.0157	1.05E+11	0.0071	0.0051	0.0122
210	103.5	0.0157	1.05E+11	0.0072	0.0052	0.0124
215	106.0	0.0158	1.06E+11	0.0072	0.0053	0.0125
220	108.5	0.0159	1.06E+11	0.0072	0.0053	0.0126
225	111.0	0.0159	1.06E+11	0.0073	0.0054	0.0127
230	113.5	0.0160	1.06E+11	0.0073	0.0055	0.0128
235	116.0	0.0161	1.06E+11	0.0073	0.0056	0.0129
240	118.5	0.0161	1.06E+11	0.0074	0.0056	0.0130
245	121.0	0.0162	1.06E+11	0.0074	0.0057	0.0131
250	123.5	0.0163	1.07E+11	0.0074	0.0058	0.0132
255	126.0	0.0163	1.06E+11	0.0075	0.0059	0.0134

TRANSNUCLEAR, INC.

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 8 OF 20  
 CALC NO 1095-16  
 REV. 0

Total Heat Transfer Coefficients, Cask Radial Surfaces

Mat, 1

$\epsilon = 0.587$

View Factor = 1.0

L = 89.00"

$T_s$ (°F)	$T_{avg}$ (°F)	k (Btu/h-ft-°F)	Ra (---)	$H_c$ (Btu/h-ft <sup>2</sup> -°F)	$H_r$ (Btu/h-in <sup>2</sup> -°F)	$H_{tot}$ (Btu/h-in <sup>2</sup> -°F)
260	128.5	0.0164	1.06E+11	0.0075	0.0060	0.0135
265	131.0	0.0164	1.06E+11	0.0075	0.0061	0.0136
270	133.5	0.0165	1.06E+11	0.0075	0.0062	0.0137
275	136.0	0.0165	1.06E+11	0.0076	0.0062	0.0138
280	138.5	0.0166	1.06E+11	0.0076	0.0063	0.0139
285	141.0	0.0167	1.06E+11	0.0076	0.0064	0.0140
290	143.5	0.0167	1.06E+11	0.0076	0.0065	0.0141
295	146.0	0.0168	1.05E+11	0.0076	0.0066	0.0142
300	148.5	0.0168	1.05E+11	0.0077	0.0067	0.0143
305	151.0	0.0169	1.05E+11	0.0077	0.0068	0.0145
310	153.5	0.0169	1.05E+11	0.0077	0.0069	0.0146
315	156.0	0.0170	1.04E+11	0.0077	0.0070	0.0147
320	158.5	0.0171	1.04E+11	0.0077	0.0071	0.0148
325	161.0	0.0171	1.04E+11	0.0078	0.0072	0.0149
330	163.5	0.0172	1.03E+11	0.0078	0.0073	0.0150
335	166.0	0.0173	1.03E+11	0.0078	0.0074	0.0152
340	168.5	0.0173	1.03E+11	0.0078	0.0075	0.0153
345	171.0	0.0174	1.02E+11	0.0078	0.0076	0.0154
350	173.5	0.0175	1.02E+11	0.0079	0.0077	0.0155
355	176.0	0.0175	1.01E+11	0.0079	0.0078	0.0156
360	178.5	0.0176	1.01E+11	0.0079	0.0079	0.0158
365	181.0	0.0176	1.01E+11	0.0079	0.0080	0.0159
370	183.5	0.0177	1.00E+11	0.0079	0.0081	0.0160
375	186.0	0.0178	9.98E+10	0.0079	0.0082	0.0161
380	188.5	0.0178	9.93E+10	0.0080	0.0083	0.0162
385	191.0	0.0179	9.89E+10	0.0080	0.0084	0.0164
390	193.5	0.0179	9.84E+10	0.0080	0.0085	0.0165
395	196.0	0.0180	9.80E+10	0.0080	0.0086	0.0166
400	198.5	0.0180	9.75E+10	0.0080	0.0087	0.0167
405	201.0	0.0181	9.70E+10	0.0080	0.0088	0.0169

TRANSNUCLEAR, INC.

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 9 OF 20  
 CALC. NO 1095-16  
 REV. 0

Total Heat Transfer Coefficients, Cask Vertical Surfaces

Mat, 2  
 $\epsilon = 0.587$   
 View Factor = 1.0  
 L = 89.00"

$T_s$ (°F)	$T_{avg}$ (°F)	K (Btu/h-ft-°F)	Ra (---)	$H_c$ (Btu/h-in <sup>2</sup> -°F)	$H_r$ (Btu/h-in <sup>2</sup> -°F)	$H_{tot}$ (Btu/h-in <sup>2</sup> -°F)
110	53.5	0.0145	8.61E+10	6.16E-03	3.83E-03	9.99E-03
115	56.0	0.0146	8.79E+10	6.22E-03	3.89E-03	1.01E-02
120	58.5	0.0146	8.96E+10	6.29E-03	3.95E-03	1.02E-02
125	61.0	0.0147	9.12E+10	6.35E-03	4.01E-03	1.04E-02
130	63.5	0.0147	9.27E+10	6.41E-03	4.07E-03	1.05E-02
135	66.0	0.0148	9.41E+10	6.46E-03	4.14E-03	1.06E-02
140	68.5	0.0148	9.54E+10	6.52E-03	4.20E-03	1.07E-02
145	71.0	0.0149	9.66E+10	6.57E-03	4.27E-03	1.08E-02
150	73.5	0.0150	9.77E+10	6.62E-03	4.33E-03	1.10E-02
155	76.0	0.0150	9.87E+10	6.67E-03	4.40E-03	1.11E-02
160	78.5	0.0151	9.96E+10	6.72E-03	4.46E-03	1.12E-02
165	81.0	0.0152	1.00E+11	6.77E-03	4.53E-03	1.13E-02
170	83.5	0.0152	1.01E+11	6.82E-03	4.60E-03	1.14E-02
175	86.0	0.0153	1.02E+11	6.86E-03	4.67E-03	1.15E-02
180	88.5	0.0154	1.03E+11	6.91E-03	4.74E-03	1.16E-02
185	91.0	0.0154	1.03E+11	6.95E-03	4.81E-03	1.18E-02
190	93.5	0.0155	1.04E+11	6.99E-03	4.88E-03	1.19E-02
195	96.0	0.0155	1.04E+11	7.03E-03	4.96E-03	1.20E-02
200	98.5	0.0156	1.05E+11	7.07E-03	5.03E-03	1.21E-02
205	101.0	0.0157	1.05E+11	7.11E-03	5.10E-03	1.22E-02
210	103.5	0.0157	1.05E+11	7.14E-03	5.18E-03	1.23E-02
215	106.0	0.0158	1.06E+11	7.18E-03	5.26E-03	1.24E-02
220	108.5	0.0159	1.06E+11	7.21E-03	5.33E-03	1.25E-02
225	111.0	0.0159	1.06E+11	7.25E-03	5.41E-03	1.27E-02
230	113.5	0.0160	1.06E+11	7.28E-03	5.49E-03	1.28E-02
235	116.0	0.0161	1.06E+11	7.31E-03	5.57E-03	1.29E-02
240	118.5	0.0161	1.06E+11	7.34E-03	5.65E-03	1.30E-02
245	121.0	0.0162	1.06E+11	7.37E-03	5.73E-03	1.31E-02
250	123.5	0.0163	1.07E+11	7.40E-03	5.81E-03	1.32E-02
255	126.0	0.0163	1.06E+11	7.43E-03	5.90E-03	1.33E-02

TRANSNUCLEAR, INC.

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 10 OF 20  
 CALC. NO 1095-16  
 REV. 0

Total Heat Transfer Coefficients, Cask Vertical Surfaces

Mat, 2

$\epsilon = 0.587$

View Factor = 1.0

L = 89.00"

$T_s$ (°F)	$T_{avg}$ (°F)	K (Btu/h-ft-°F)	Ra (---)	$H_c$ (Btu/h-ft <sup>2</sup> -°F)	$H_r$ (Btu/h-in <sup>2</sup> -°F)	$H_{tot}$ (Btu/h-in <sup>2</sup> -°F)
260	128.5	0.0164	1.06E+11	7.45E-03	5.98E-03	1.34E-02
265	131.0	0.0164	1.06E+11	7.48E-03	6.07E-03	1.35E-02
270	133.5	0.0165	1.06E+11	7.50E-03	6.15E-03	1.37E-02
275	136.0	0.0165	1.06E+11	7.52E-03	6.24E-03	1.38E-02
280	138.5	0.0166	1.06E+11	7.54E-03	6.33E-03	1.39E-02
285	141.0	0.0167	1.06E+11	7.56E-03	6.42E-03	1.40E-02
290	143.5	0.0167	1.06E+11	7.58E-03	6.50E-03	1.41E-02
295	146.0	0.0168	1.05E+11	7.60E-03	6.60E-03	1.42E-02
300	148.5	0.0168	1.05E+11	7.62E-03	6.69E-03	1.43E-02
305	151.0	0.0169	1.05E+11	7.64E-03	6.78E-03	1.44E-02
310	153.5	0.0169	1.05E+11	7.66E-03	6.87E-03	1.45E-02
315	156.0	0.0170	1.04E+11	7.68E-03	6.97E-03	1.47E-02
320	158.5	0.0171	1.04E+11	7.71E-03	7.06E-03	1.48E-02
325	161.0	0.0171	1.04E+11	7.73E-03	7.16E-03	1.49E-02
330	163.5	0.0172	1.03E+11	7.75E-03	7.26E-03	1.50E-02
335	166.0	0.0173	1.03E+11	7.77E-03	7.36E-03	1.51E-02
340	168.5	0.0173	1.03E+11	7.79E-03	7.46E-03	1.52E-02
345	171.0	0.0174	1.02E+11	7.80E-03	7.56E-03	1.54E-02
350	173.5	0.0175	1.02E+11	7.82E-03	7.66E-03	1.55E-02
355	176.0	0.0175	1.01E+11	7.84E-03	7.76E-03	1.56E-02
360	178.5	0.0176	1.01E+11	7.86E-03	7.86E-03	1.57E-02
365	181.0	0.0176	1.01E+11	7.87E-03	7.97E-03	1.58E-02
370	183.5	0.0177	1.00E+11	7.89E-03	8.07E-03	1.60E-02
375	186.0	0.0178	9.98E+10	7.90E-03	8.18E-03	1.61E-02
380	188.5	0.0178	9.93E+10	7.91E-03	8.29E-03	1.62E-02
385	191.0	0.0179	9.89E+10	7.92E-03	8.40E-03	1.63E-02
390	193.5	0.0179	9.84E+10	7.94E-03	8.51E-03	1.64E-02
395	196.0	0.0180	9.80E+10	7.95E-03	8.62E-03	1.66E-02
400	198.5	0.0180	9.75E+10	7.96E-03	8.73E-03	1.67E-02
405	201.0	0.0181	9.70E+10	7.97E-03	8.84E-03	1.68E-02

TRANSNUCLEAR, INC.

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 11 OF 20  
 CALC. NO 1095-16  
 REV. 0

Air Properties

Temperature <sup>(1)</sup> (K)	$\rho^{(1)}$ (kg/m <sup>3</sup> )	Conductivity <sup>(1)</sup> (W/m-K)	Prandtl # <sup>(1)</sup> (-----)	Dyn. Visc. <sup>(1)</sup> (Pa-s)
255	1.386	22.68E-3	0.721	16.25E-6
265	1.333	23.48E-3	0.717	16.75E-6
280	1.261	24.67E-3	0.713	17.50E-6
295	1.197	25.85E-3	0.709	18.22E-6
310	1.139	27.01E-3	0.705	18.93E-6
325	1.086	28.15E-3	0.702	19.63E-6
340	1.038	29.28E-3	0.699	20.30E-6
355	0.9945	30.39E-3	0.696	20.97E-6
370	0.9539	31.50E-3	0.693	21.60E-6
385	0.9169	32.59E-3	0.690	22.24E-6
400	0.8822	33.65E-3	0.689	22.86E-6
420	0.8402	35.05E-3	0.687	23.66E-6
440	0.8021	36.43E-3	0.684	24.45E-6

Temperature (°F)	Conductivity (Btu/hr-ft-°F)	Conductivity (Btu/hr-in-°F)	Prandtl # (-----)	Kin. Visc. (ft <sup>2</sup> -hr)
-1	0.0131	0.0011	0.721	0.4543
17	0.0136	0.0011	0.717	0.4869
44	0.0143	0.0012	0.713	0.5378
71	0.0149	0.0012	0.709	0.5898
98	0.0156	0.0013	0.705	0.6440
125	0.0163	0.0014	0.702	0.7004
152	0.0169	0.0014	0.699	0.7578
179	0.0176	0.0015	0.696	0.8171
206	0.0182	0.0015	0.693	0.8775
233	0.0188	0.0016	0.690	0.9399
260	0.0194	0.0016	0.689	1.0041
296	0.0203	0.0017	0.687	1.0912
332	0.0210	0.0018	0.684	1.1812

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 12 OF 20  
CALC. NO 1095-16  
REV. 0

HTOT\_HCY.MAC

**PROPRIETARY INFORMATION WITHHELD UNDER  
10CFR 2.390**

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 13 OF 20  
CALC. NO 1095-16  
REV. 0

**PROPRIETARY INFORMATION WITHHELD UNDER  
10CFR 2.390**

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 14 OF 20  
CALC NO 1095-16  
REV. 0

**PROPRIETARY INFORMATION WITHHELD UNDER  
10CFR 2.390**



TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 15 OF 20  
CALC. NO 1095-16  
REV. 0

HTOT\_VPL.MAC

**PROPRIETARY INFORMATION WITHHELD UNDER  
10CFR 2.390**

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 16 OF 20  
CALC. NO 1095-16  
REV. 0

**PROPRIETARY INFORMATION WITHHELD UNDER  
10CFR 2.390**

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 17 OF 20  
CALC. NO 1095-16  
REV. 0

**PROPRIETARY INFORMATION WITHHELD UNDER  
10CFR 2.390**

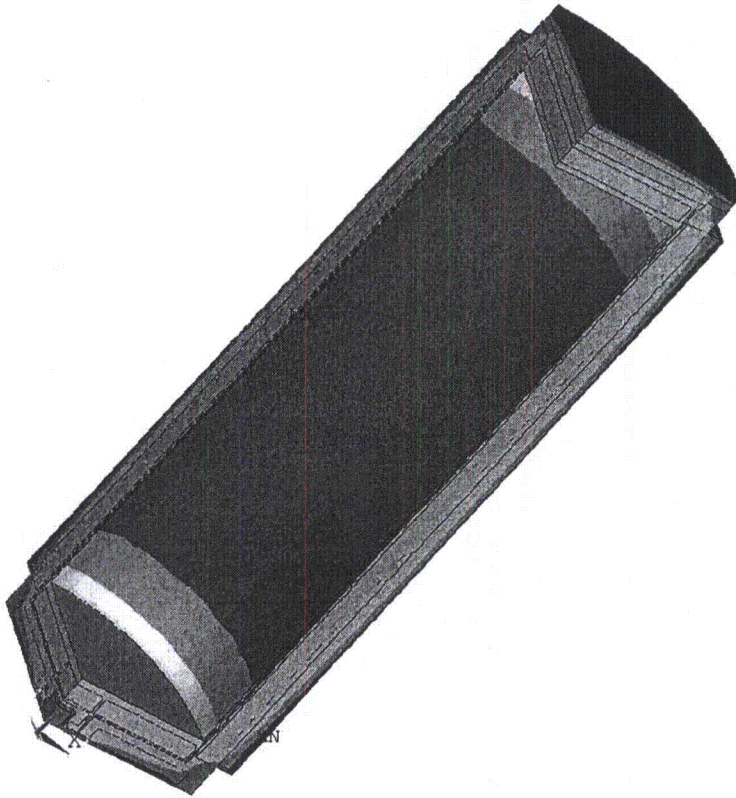
TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 18 OF 20  
CALC NO 1095-16  
REV. 0

## APPENDIX B

### Temperature Distributions

#### Cask Body Model

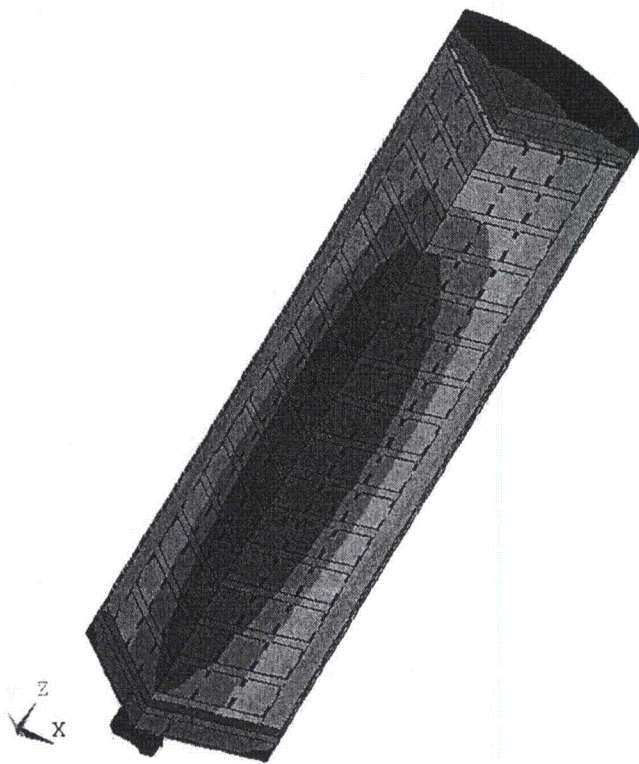


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APR 2 2001  
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SUB =1  
TIME=1  
TEMP (AVG)  
RSYS=0  
PowerGraphics  
EFACET=1  
AVRES=Mat  
SMN =81.593  
SMX =370.792  
81.593  
113.727  
145.86  
177.993  
210.126  
242.26  
274.393  
306.526  
338.659  
370.792

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 19 OF 20  
CALC. NO 1095-16  
REV. 0

Basket Model

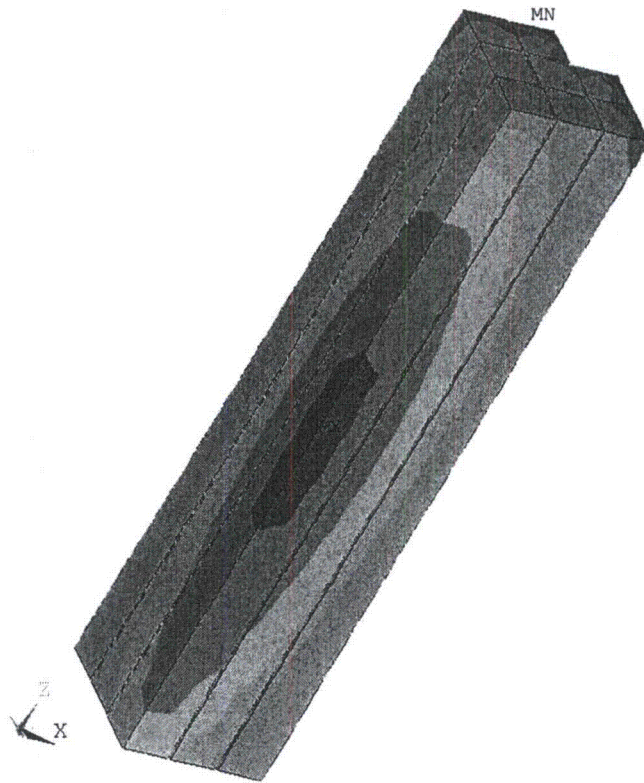


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TIME=1  
TEMP (AVG)  
RSYS=0  
PowerGraphics  
EFACET=1  
AVRES=Mat  
SMN =205.964  
SMX =638.173  
205.964  
253.988  
302.011  
350.034  
398.057  
446.08  
494.103  
542.126  
590.15  
638.173

TITLE NUHOMS-32P,  
Transfer Thermal Analysis,  
-3 °F Ambient

SHEET 20 OF 20  
CALC. NO 1095-16  
REV. 0

Fuel Assemblies



ANSYS 5.6  
APR 2 2001  
10:36:27  
NODAL SOLUTION  
STEP=1  
SUB =1  
TIME=1  
TEMP  
SMN =394.117  
SMX =663.536  
394.117  
424.052  
453.988  
483.923  
513.859  
543.794  
573.73  
603.665  
633.601  
663.536