

## 1.0 OBJECTIVE

Perform the structural analyses of the EnergySolutions 3-60B Cask under drop conditions, using a 3-dimensional finite element model.

## 2.0 INTRODUCTION

EnergySolutions 3-60B Cask (Reference 1) is designed as a Type B radioactive-material shipping package. To be certified by the U.S.N.R.C., the cask needs to meet the requirements of 10 CFR 71 (Reference 2) and follow the guidelines of U.S.N.R.C. Regulatory Guide 7.8 (Ref. 3).

This document presents the structural analysis of the 3-60B Cask under various drop conditions required by the code. The analyses in this document are performed using the finite element modeling techniques. A three-dimensional model of the cask that includes all its major components has been employed in the analyses. Temperature dependent material properties of the major components of the cask are used in the analyses.

Analyses of the 3-60B Cask package have been performed for hypothetical accident condition (HAC) and normal condition of transport (NCT) drop test using the methodology developed by EnergySolutions. The details of the analyses are documented in the proprietary document of Reference 4. The resultant impact loads during various drop tests are obtained from this document and applied to the detailed finite element model of the cask body. Every component of the cask is evaluated for its integrity during the drop tests by comparing the stress intensities with their corresponding allowable values.

The results of the analyses for various load cases are presented pictorially in stress intensity contour plots as well as in table form; with the corresponding safety factors in each component of the cask body.

## 3.0 REFERENCES

1. EnergySolutions Drawing No. C-002-165024-001, Rev.0, 3-60B Cask General Arrangement and Details.
2. Code of Federal Regulations, Title 10, Part 71, Packaging and Transportation of Radioactive Material, January 2003.
3. U.S. NRC Regulatory Guide 7.8, Revision 1, March 1989, Load Combinations for the Structural Analysis of Shipping Casks for Radioactive Material.
4. EnergySolutions Document No. ST-557, Rev.1, Drop Analyses of the 3-60B Cask Package Using LS-DYNA Program.
5. ASME Boiler & Pressure Vessel Code, Section II, Part D, Materials, The American Society of Mechanical Engineers, New York, NY, 2005.

6. NUREG 0481/SAND77-1872, An Assessment of Stress-Strain Data Suitable for Finite Element Elastic-Plastic Analysis of Shipping Containers, Sandia National Laboratories, 1978.
7. U.S. NRC Regulatory Guide 7.6, Revision 1, Design Criteria for the Structural Analysis of Shipping Cask Containment Vessels, 1978.
8. ANSYS, Rev. 11.0, Computer Software, ANSYS Inc., Canonsburg, PA, 2007.
9. *EnergySolutions* Document No. ST-501, Rev.1, Structural Analyses of the 3-60B Cask under Normal Conditions of Transport.
10. *EnergySolutions* Document TH-022, Rev. 2, Steady State Thermal Analyses of the 3-60B Cask Using a 3-D Finite Element Model.
11. SAND88-0616-UC-71, Numerical and Analytical Methods for Approximating the Eccentric Impact Response (Slapdown) of Deformable Bodies, Sandia National Laboratories, 1988.

**4.0 MATERIAL PROPERTIES**

Material	Temp. (°F)	Strength (ksi)			Young's Modulus (10 <sup>6</sup> psi)	Coefficient of Thermal Expansion (10 <sup>-6</sup> in/in)
		Yield (S <sub>y</sub> )	Ultimate (S <sub>u</sub> )	Membrane Allowable (S <sub>m</sub> )		
ASTM A240 Type 304L		(1)	(1)	(1)	(1)	(1)
	-20	25.0	70.0	16.7	28.8	-
	70	25.0	70.0	16.7	28.3	8.5
	100	25.0	70.0	16.7	-	8.6
	200	21.4	66.1	16.7	27.5	8.9
	300	19.2	61.2	16.7	27.0	9.2
	400	17.5	58.7	15.8	26.4	9.5
500	16.4	57.5	14.7	25.9	9.7	
ASTM A240 Gr. 45 & ASTM A182 Gr. F45		(1)	(1)	(1)	(1)	(1)
	-20	45.0	87.0	24.9	28.8	-
	70	45.0	87.0	24.9	28.3	8.5
	100	45.0	87.0	24.9	-	8.6
	200	37.5	86.4	24.7	27.5	8.9
	300	33.0	81.6	23.3	27.0	9.2
	400	29.9	78.5	22.4	26.4	9.5
500	27.8	76.4	21.8	25.9	9.7	
ASTM A354 Gr. BD (Lid Bolts)		(1)	(1)	(1)	(1)	(1)
	-20	130	150	30	29.7	-
	70	130	150	30	29.2	6.4
	100	130	150	30	-	6.5
	200	119.1	150	30	28.6	6.7
	300	115	150	30	28.1	6.9
	400	111	150	30	27.7	7.1
500	105.9	150	30	27.1	7.3	
ASTM B29 Lead		(2)			(2)	(2)
	-20	-	-	-	2.43	15.65
	70	5	-	-	2.27	16.06
	100	-	-	-	2.21	16.22
	200	-	-	-	2.01	16.70
	300	-	-	-	1.85	17.33
	400	-	-	-	1.70	18.16
500	-	-	-	1.52	19.12	

Notes:

- (1) From ASME B&PV Code 2004, Section II, Part D (Reference 5).
- (2) From NUREG/CR 0481 (Reference 6)

**5.0 ALLOWABLE STRESSES**

Material →		ASTM A240 Type 304L	ASTM A182 Gr.F45 & A240 Gr. 45	ASTM A354 Gr. BD
Yield Stress, $S_y$	(psi)	25,000 <sup>(1)</sup>	45,000 <sup>(1)</sup>	130,000 <sup>(1)</sup>
Ultimate Stress, $S_u$	(psi)	70,000 <sup>(1)</sup>	87,000 <sup>(1)</sup>	150,000 <sup>(1)</sup>
Design Stress Intensity, $S_m$	(psi)	16,700 <sup>(1)</sup>	24,900 <sup>(1)</sup>	30,000 <sup>(1)</sup>
Normal Conditions	Membrane Stress	16,700 <sup>(2)</sup>	24,900 <sup>(2)</sup>	60,000 <sup>(2)</sup>
	Mem. + Bending Stress	25,050 <sup>(2)</sup>	37,350 <sup>(2)</sup>	90,000 <sup>(2)</sup>
	Peak Stress	50,100 <sup>(3)</sup>	74,700 <sup>(3)</sup>	150,000 <sup>(3)</sup>
Hypothetical Accident Conditions	Membrane Stress	40,080 <sup>(4)</sup>	59,760 <sup>(4)</sup>	105,000 <sup>(4)</sup>
	Mem. + Bending Stress	60,120 <sup>(4)</sup>	87,000 <sup>(4)</sup>	150,000 <sup>(4)</sup>
	Peak Stress	140,000 <sup>(5)</sup>	174,000 <sup>(5)</sup>	300,000 <sup>(5)</sup>

## Notes:

- (1) From ASME B&PV Code 2004, Section II, Part D (Reference 5).
- (2) Established from Regulatory Guide 7.6 (Reference 7).
- (3) Established from Regulatory Guide 7.6, Regulatory Position 4, and ASME, Section III, Division 3, WB-3200 criteria. The limit on this stress component is  $3S_m$ .
- (4) Regulatory Guide 7.6 (Reference 7) does not provide any criteria. ASME B&PV Code, Section III, Appendix F has been used to establish these criteria.
- (5) Regulatory Guide 7.6, Regulatory Position 7 and ASME Section III, Division 3, WB-3221.9 criteria of limiting these stresses to  $2S_a$  @ 10 cycles results in higher than  $2S_u$  allowable values. The limits for peak stresses are conservatively set to be  $2S_u$ .

**6.0 MODEL DESCRIPTION**

The structural analyses of the 3-60B Cask under various drop test conditions have been performed using finite element modeling techniques. ANSYS finite element analysis code (Ref. 8) has been employed to perform the analyses. Since for all the drop orientations (end,



side, corner, and slap-down), at least one plane of symmetry exists, a 180° model has been employed in all the analyses. This model has been developed from the 11.25° model developed in References 9 and 10 for the structural and thermal analyses of the cask during normal conditions of transport.

The model of the cask is made using 3-dimensional 8-node structural solid elements (ANSYS SOLID185) to represent the major components of the cask, the cask body, the lid, and the bolts. The fire shield does not provide any structural strength to the cask. Therefore, it is not included in the model.

The poured lead in the body is not bonded to the steel. It is free to slide over the steel surface. Therefore, the interface between the lead and the steel is modeled by pairs of 3-d 8 node contact element (CONTA174) and 3-d target segment (TARGE170) elements. These elements allow the lead to slide over the steel at the same time prevent it from penetrating the steel surface. The interface between the two plates that form the lid is also modeled by the contact-target pairs. The transition from a coarser mesh to a finer mesh, as well as bondage between various parts of the model, is also modeled using these elements.

Figure 1 shows the outline of the model depicting the material numbering. Figure 2 shows the finite element grid of the lid, seal plate, and the bolts. Figure 3 shows the finite element grid of the cask body without the lead and Figure 4 shows that of the lead. The interface between various components of the cask is modeled by target-contact surface definition. Figure 5 shows target surfaces of various contact-target pairs. The printout of the pertinent model quantities is included in Appendix 1.

### **Boundary Conditions**

Since the model of the cask includes 180° geometry, symmetry boundary conditions are used on the cut-plane of the model in all the analyses. Also, the rigid body motion is prevented in the model by restraining it at the locations where such restraints have insignificant effect on the overall behavior of the model. This is necessary since the quasi-static analyses performed for every drop condition will result in a small net force in the plane of symmetry that will give rise to a rigid body motion.

### **Loading**

Applied loading is described for each drop orientation under the corresponding analysis section.

### **Temperature**

The temperature distribution under various drop conditions is obtained from the thermal analyses performed in Reference 10 and is applied as the nodal temperature in the finite element model.

### Internal Pressure

The cask internal pressure of 35 psig is applied over the nodes representing the cavity of the cask under various drop conditions in the hot environment. No internal pressures are applied during all drop conditions in the cold environment, with or without the internal decay heat.

### Inertia Load

Cask body inertia, under various drop conditions, is applied as a body load. The magnitude of the inertia load is given in the corresponding analysis section. It should be noted that because of the segmentation of arc length in the finite element models, the mass of the model is always lower than the actual mass. To account for this, as well as to include the mass of miscellaneous items not included in the model, an adjustment is made in the value of acceleration due to gravity.

$$\text{Cask Body Mass} = 80,000 - 9,500 - 2 \times 3,800 = 62,900 \text{ lb}$$

$$\text{Mass of the FEM} = 2 \times 28,409 = 56,818 \text{ lb}$$

$$\text{Acceleration multiplication factor} = 62,900/56,818 = 1.107$$

## 7.0 ANALYSES

The finite element model (FEM) described above is analyzed for the accelerations obtained from the EnergySolutions proprietary analyses documented in Reference 4. The distribution of various loading components is described in details in the following sections.

### 7.1 HAC Drop Tests

#### 7.1.1 End Drop

The following impact limiter reactions are obtained from Reference 4.

$$\text{Cold Conditions} = 3.954 \times 10^6 \text{ lb} \quad (\text{Table 3 and Figure 56 of Reference 4})$$

$$\text{Hot Conditions} = 3.083 \times 10^6 \text{ lb} \quad (\text{Table 3 and Figure 61 of Reference 4})$$

Conservatively use the maximum of the two reactions for the analyses of all environmental conditions. The impact limiter reaction is converted to the rigid body acceleration by dividing the reaction by that portion of the mass of the package which causes this reaction. During the end drop test the impact limiter reaction is caused by the total mass of the package less the mass of one impact limiter, i.e.  $80,000 - 3,800 = 76,200 \text{ lb}$  (Reference 3-60B SAR Section 2.1.3). Since the FEM represents only  $\frac{1}{2}$  of the package, the total mass is divided by 2 in the calculation of the rigid body acceleration. A factor of 1.1 is used to conservatively increase this reaction in the analyses.

$$\text{Rigid body acceleration} = 1.1 \times 2 \times 3.954 \times 10^6 / 76,200 = 114.2 \gg \text{Use } 150g$$

For the quasi-static analysis of the cask under end drop test conditions the inertia loads and reactions are distributed to the cask body as shown in Figure 6.

#### Impact limiter Inertia

The inertia load of the lower impact limiter is included as the uniform pressure on the surface where the impact limiter contacts the cask.

$$\text{Mass of each Impact Limiter} = 3,800 \text{ lb}$$

$$\text{Inside Radius of the Impact Limiter} = 12 \text{ in (nearest node in the FEM is at 12.5 in)}$$

$$\text{Outside Radius of the Cask} = 25.5 \text{ in}$$

Pressure on the cask due to impact limiter inertia,

$$p_{LL} = 150 \times 3,800 / [\pi \times (25.5^2 - 12.5^2)] = 367.3 \text{ psi}$$

#### Payload Inertia

The payload inertia is applied as a uniform surface pressure over the lid inside surface. The lid has a radius of 17.375 in. For 9,500 lb total mass of payload, the magnitude of the pressure is:

$$p_{lid} = 150 \times 9,500 / (\pi \times 17.375^2) = 1,502.5 \text{ psi}$$

#### Cask Body Inertia

Cask body inertia is applied as the body force. As explained earlier, to account for the total mass of the package a factor of 1.107 is used to increase the FEM mass.

$$\text{Cask Body Acceleration} = 150 \times 1.107 = 166 \text{ g}$$

#### Impact Limiter Reaction

The impact limiter reaction is simulated by restraining the nodes at the impact limiter-lid interface in vertical direction.

#### Model Analyses

The FEM is analyzed under the above loading for hot (Load Case 1), cold with maximum decay heat (Load Case 2), and cold with no decay heat, environmental conditions. The cask body temperature for Load Cases 1 and 2 are obtained from the hot and cold environmental loading conditions of Reference 10. For Load Case 3 (no internal heat), the entire cask is assumed to be at -20°F. Figures 7 through 9 show the cask body temperature profile and the pressure distributions used in the FEM for Load Cases 1 through 3.

Figures 10 through 12 show the stress intensity plot in the cask body for the three load cases. Stress intensities are calculated in each major component of the cask and are presented in Tables 1 through 3. These tables also categorize the stresses based on the ASME code and compare them with the corresponding allowable value established in Section 5.0. Factors of safety based on the ratio of allowable stress to the calculated stress are also calculated.

### 7.1.2 Side Drop

The following impact limiter reactions are obtained from Reference 4.

Cold Conditions =  $1.889 \times 10^6$  lb (Table 3 and Figure 66 of Reference 4)

Hot Conditions =  $1.636 \times 10^6$  lb (Table 3 and Figure 71 of Reference 4)

Conservatively use the maximum of the two reactions for the analyses of all environmental conditions. The impact limiter reaction is converted to the rigid body acceleration by dividing the reaction by that portion of the mass of the package which causes this reaction. During the side drop test the impact limiter reaction is caused by the total mass of the package less the mass of the two impact limiters, i.e.  $80,000 - 2 \times 3,800 = 72,400$  lb (Reference 3-60B SAR Section 2.1.3). Since the FEM represents only  $\frac{1}{2}$  of the package and each impact limiter reaction is caused by  $\frac{1}{2}$  the participating mass, the total mass is divided by 4 in the calculation of the rigid body acceleration. A factor of 1.1 is used to conservatively increase this reaction in the analyses.

$$\text{Rigid body acceleration} = 1.1 \times 4 \times 1.889 \times 10^6 / 72,400 = 114.8 \gg \text{Use } 120g$$

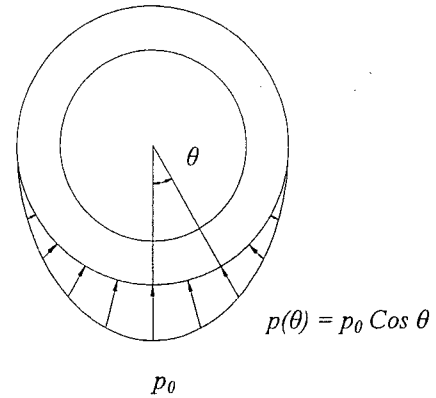
For the quasi-static analysis of the cask under side drop test conditions reactions and the inertia loads are distributed to the cask body as shown in Figure 13.

#### Impact Limiter Reactions

The impact limiter reactions are applied as surface pressure on the lower half of the impact limiter-cask interface. This pressure is assumed to be uniform along the axis of the cask but varies sinusoidally along the circumference. For such a distribution, the following mathematical derivation is used:

$$p(\theta) = p_0 \cos \theta \quad -\pi/2 \leq \theta \leq \pi/2$$

$$\begin{aligned} F_v &= \int_{-\pi/2}^{\pi/2} p_0 \cdot \cos \theta \cdot r \cdot d\theta \cdot \cos \theta \\ &= p_0 \cdot r \int_{-\pi/2}^{\pi/2} \cos^2 \theta \cdot d\theta \\ &= p_0 \cdot r \int_{-\pi/2}^{\pi/2} \frac{1}{2} \cdot (1 + \cos 2\theta) \cdot d\theta \\ &= \frac{p_0 \cdot r}{2} \left[ \theta + \frac{\sin 2\theta}{2} \right]_{-\pi/2}^{\pi/2} \\ &= \frac{\pi \cdot r \cdot p_0}{2} \end{aligned}$$



$$\begin{aligned} F_h &= \int_{-\pi/2}^{\pi/2} p_0 \cdot \cos \theta \cdot r \cdot d\theta \cdot \sin \theta \\ &= p_0 \cdot r \int_{-\pi/2}^{\pi/2} \frac{1}{2} \cdot \sin 2\theta \cdot d\theta \\ &= \frac{p_0 \cdot r}{2} \left[ \frac{-\cos 2\theta}{2} \right]_{-\pi/2}^{\pi/2} \\ &= 0 \end{aligned}$$

Reaction of the cask at the two impact limiter locations,  $2R = (80,000 - 2 \times 3,800) \times 120$

Reaction at each impact limiter location,  $R = \frac{1}{2} \times (80,000 - 2 \times 3,800) \times 120 = 4.344 \times 10^6$  lb

The top impact limiter reaction is applied at the surface that has a radius of 25.5 in and extends in the axial direction over a length of 20.28 in. Thus,

$$p_0 = (2 \times 4.344 \times 10^6) / (\pi \times 25.5 \times 20.28) = 5,348 \text{ psi}$$

The bottom impact limiter reaction is applied at the surface that has a radius of 25.5 in and extends in the axial direction over a length of 21.355 in. Thus,

$$p_0 = (2 \times 4.344 \times 10^6) / (\pi \times 25.5 \times 21.355) = 5,078 \text{ psi}$$

### Payload Inertia

The payload inertia load is applied as surface pressure on the lower half of the inner shell of the cask. This pressure is assumed to be uniform along the axis of the cask but varies

sinusoidally along the circumference. The radius of the inner shell is 17.5 in and its length is 109 inch. Thus,

$$p_0 = (2 \times 120 \times 9,500) / (\pi \times 17.5 \times 109) = 380.5 \text{ psi}$$

### Cask Body Inertia

Cask body inertia is applied as the body force. As explained earlier, to account for the total mass of the package a factor of 1.107 is used to increase the FEM mass.

$$\text{Cask Body Acceleration} = 120 \times 1.107 = 132.84 \text{ g}$$

### Model Analyses

The FEM is analyzed under the above loading for hot (Load Case 1), cold with maximum decay heat (Load Case 2), and cold with no decay heat, environmental conditions. The cask body temperature for Load Cases 1 and 2 are obtained from the hot and cold environmental loading conditions of Reference 10. For Load Case 3 (no internal heat), the entire cask is assumed to be at -20°F. Figures 14 through 16 show the cask body temperature profile and the pressure distributions used in the FEM for Load Cases 1 through 3.

Figures 17 through 19 show the stress intensity plot in the cask body for the three load cases. Stress intensities are calculated in each major component of the cask and are presented in Tables 4 through 6. These tables also categorize the stresses based on the ASME code and compare them with the corresponding allowable value established in Section 5.0. Factors of safety based on the ratio of allowable stress to the calculated stress are also calculated.

### **7.1.3 Corner Drop**

The following impact limiter reactions are obtained from Reference 4.

$$\text{Cold Conditions} = 2.080 \times 10^6 \text{ lb} \quad (\text{Table 3 and Figure 76 of Reference 4})$$

$$\text{Hot Conditions} = 1.847 \times 10^6 \text{ lb} \quad (\text{Table 3 and Figure 81 of Reference 4})$$

Conservatively use the maximum of the two reactions for the analyses of all environmental conditions. The impact limiter reaction is converted to the rigid body acceleration by dividing the reaction by that portion of the mass of the package which causes this reaction. During the corner drop test the impact limiter reaction is caused by the total mass of the package less the mass of one impact limiter, i.e.  $80,000 - 3,800 = 76,200 \text{ lb}$  (Reference 3-60B SAR Section 2.1.3). Since the FEM represents only  $\frac{1}{2}$  of the package, the total mass is divided by 2 in the calculation of the rigid body acceleration. A factor of 1.1 is used to conservatively increase this reaction in the analyses.

$$\text{Rigid body acceleration} = 1.1 \times 2 \times 2.080 \times 10^6 / 76,200 = 60.1 \quad \gg \text{ Use } 70\text{g}$$

For the quasi-static analysis of the cask under corner drop test conditions reactions and the inertia loads are distributed to the cask body as shown in Figure 20.

The finite element model of the cask shows that the C.G. of the cask is 65.45 in from the bottom side of the lid. The cask has a radius of 25.5 in. Therefore, for the C.G. to be directly above the corner, the cask axis will be inclined from the vertical axis by an angle,

$$\alpha = \tan^{-1}(25.5/65.45) = 21.29^\circ$$

Thus, the axial acceleration,

$$g_a = 70 \times \cos 21.29^\circ = 65.22 \text{ g}$$

Lateral acceleration,

$$g_l = 70 \times \sin 21.29^\circ = 25.42 \text{ g}$$

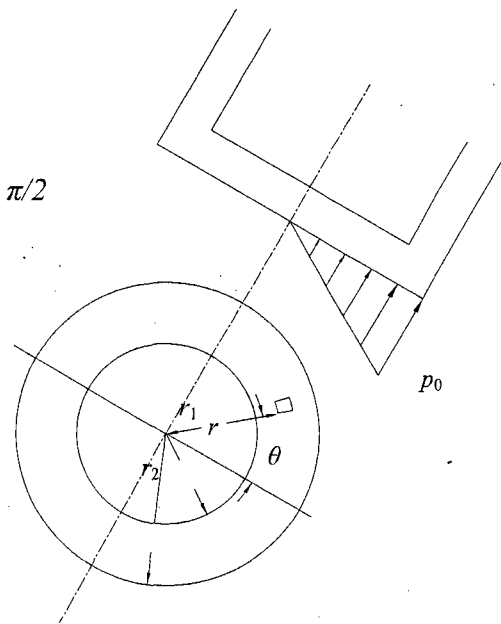
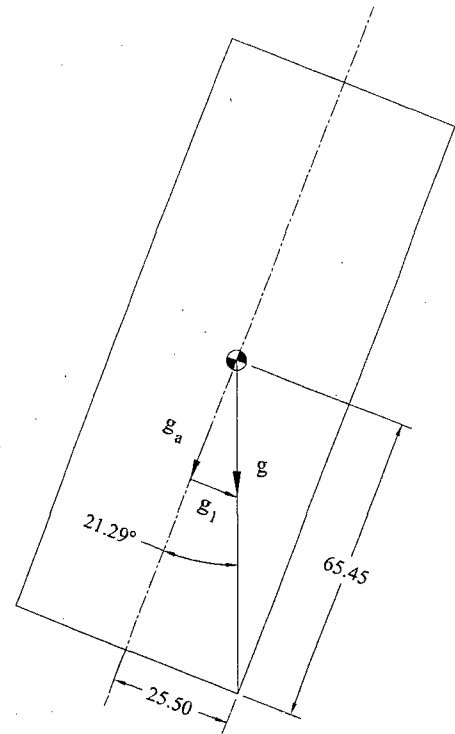
Impact Limiter Reaction

The impact limiter reaction is resolved into an axial and a lateral component. The axial component is applied to the lid surface at the interface with the impact limiter. The pressure is assumed to vary sinusoidally along the tangential direction. Mathematically this pressure may be represented by:

$$p(r, \theta) = p_0 \cdot \frac{r \cdot \cos \theta}{r_2} \cdot r \cdot d\theta \cdot dr \quad -\pi/2 \leq \theta \leq \pi/2$$

This distribution results in the total axial load, F that can be calculated by integration as follows:

$$\begin{aligned} F &= \int_{-\pi/2}^{\pi/2} \int_{r_1}^{r_2} \frac{p_0}{r_2} \cdot r \cdot \cos \theta \cdot r \cdot dr \cdot d\theta \\ &= \frac{p_0}{r_2} \cdot \int_{-\pi/2}^{\pi/2} \cos \theta \cdot d\theta \cdot \int_{r_1}^{r_2} r^2 \cdot dr \\ &= \frac{2}{3} \cdot \frac{p_0}{r_2} \cdot (r_2^3 - r_1^3) \end{aligned}$$



The axial component of the impact limiter reaction,

$$R_a = (80,000 - 3,800) \times 65.22 = 4.97 \times 10^6 \text{ lb}$$

The reaction is distributed over the lid surface annulus having an inside radius of 15 in and outside radius of 25.5 in. To get the total load  $R_a$  on the lid surface,  $p_0$  must be,

$$p_0 = 4.97 \times 10^6 \cdot \frac{3}{2} \cdot \left( \frac{25.5}{25.5^3 - 15^3} \right) = 14,395 \text{ psi}$$

Since the lid has the bolt-hole cut-outs in this region, thereby reducing the area over which this load is applied, adjustment in the above pressure value must be made. This adjustment was manually made using the FEM. To obtain the total load of  $4.97 \times 10^6$  lb, the value for  $p_0$  was increased to 17,409 psi.

The lateral component of the impact limiter reaction was applied in the manner as described under side drop loading with the exception that the magnitude of the pressure is also varied linearly from the maximum value to zero at the top of the impact limiter edge.

The lateral component of the impact limiter reaction is:

$$R_l = (80,000 - 3,800) \times 25.42 = 1.937 \times 10^6 \text{ lb}$$

This reaction is applied at the surface that has a radius of 25.5 in and extends in the axial direction over a length of 20.28 in. Following the derivation under side drop,

$$p_0 = 2 \times [(2 \times 1.937 \times 10^6) / (\pi \times 25.5 \times 20.28)] = 4,769 \text{ psi}$$

Note that a multiplier of 2 is used to account for the axial variation of the pressure.

### Impact Limiter Inertia

The upper impact limiter inertia is resolved into an axial component and a lateral component. The axial component is applied in the same manner as described under end drop and the lateral component is applied in the same manner as described for the side drop impact limiter reaction.

Magnitude of the uniform pressure representing the impact limiter axial inertia is:

$$p_{LL} = 65.22 \times 3,800 / [\pi \times (25.5^2 - 12.5^2)] = 159.7 \text{ psi}$$

Amplitude of the sinusoidally varying pressure, representing the impact limiter lateral inertia,

$$p_0 = (2 \times 25.42 \times 3,800) / (\pi \times 25.5 \times 21.355) = 112.93 \text{ psi}$$



Payload Inertia

The payload inertia is resolved into an axial component and a lateral component. The axial component is applied to the lid in the same manner as described under the end drop. The lateral component is applied to the lower half of the inner shell as described under side drop.

Magnitude of the uniform pressure representing the payload axial inertia is:

$$p_{\text{lid}} = 65.22 \times 9,500 / (\pi \times 17.375^2) = 653.3 \text{ psi}$$

Amplitude of the sinusoidally varying pressure, representing the payload lateral inertia,

$$p_0 = (2 \times 25.42 \times 9,500) / (\pi \times 17.5 \times 109) = 80.6 \text{ psi}$$

Cask Body Inertia

Cask body inertia is applied as the body force. As explained earlier, to account for the total mass of the package a factor of 1.107 is used to increase the FEM mass.

$$\text{Cask Body Axial Acceleration} = 65.22 \times 1.107 = 72.2 \text{ g}$$

$$\text{Cask Body Lateral Acceleration} = 25.42 \times 1.107 = 28.14 \text{ g}$$

Model Analyses

The FEM is analyzed under the above loading for hot (Load Case 1), cold with maximum decay heat (Load Case 2), and cold with no decay heat, environmental conditions. The cask body temperature for Load Cases 1 and 2 are obtained from the hot and cold environmental loading conditions of Reference 10. For Load Case 3 (no internal heat), the entire cask is assumed to be at -20°F. Figures 21 through 23 show the cask body temperature profile and the pressure distributions used in the FEM for Load Cases 1 through 3.

Figures 24 through 26 show the stress intensity plot in the cask body for the three load cases. Stress intensities are calculated in each major component of the cask and are presented in Tables 7 through 9. These tables also categorize the stresses based on the ASME code and compare them with the corresponding allowable value established in Section 5.0. Factors of safety based on the ratio of allowable stress to the calculated stress are also calculated.

**7.1.4 Shallow Angle Drop**

As described in the Reference 4, the 3-60B cask package has also been analyzed for shallow angle drop tests. For these tests, the cask axis makes angles of 7½° and 15° with the horizontal plane. These orientations are referred to as slapdown-1 and slapdown-2, respectively. Under both these drop conditions the node-end impact limiter makes contact with the rigid target surface first. This is followed by a rotation of the cask and the tail-end impact limiter then strikes the rigid surface. With the four orientations for the drop test addressed in this document the entire spectrum of initial orientations of the cask package for

the hypothetical drop test has been covered. The FEM analyses have performed for sufficiently large time durations in which both primary as well as secondary impacts, if any, take place. Thus, the slap-down effect of the shallow angle drop, as well as that during the corner-over-C.G. drop has been included in these analyses.

The results of the shallow angle drop analyses show that the tail-end impact is more severe than the nose-end impact for the 3-60B cask. This result is consistent with the conclusion of Reference 11, which shows that for a slender cask with length-to-radius of gyration ratio larger than 2, the tail-end impact is more severe than the nose-end impact. For both shallow angle orientations (slapdown-1 and slapdown-2), and for both cold and hot environmental conditions, the tail-end impact reactions are larger than nose-end impact limiter reactions (see Figures 86, 91, 96 and 101 of Reference 4).

The largest impact limiter reaction for the slapdown-1 and slapdown-2 and in both cold and hot environment case is:

$$R_{\text{shallow-angle}} = 2.009 \times 10^6 \text{ lb} \quad (\text{Table 3 and Figure 86 of Reference 4})$$

The nature of impact limiter reaction in this case is very similar to that of the side drop test. The maximum impact limiter reaction during the side drop test is:

$$R_{\text{side-drop}} = 1.889 \times 10^6 \text{ lb} \quad (\text{Table 3 and Figure 66 of Reference 4})$$

Thus, the shallow angle drop test will result in the impact limiter reaction that is larger than that of the side drop test by a factor of:

$$2.009/1.889 = 1.06$$

Therefore, a factor of safety of 1.06 or larger in the cask due to HAC side drop loading will ensure that cask will satisfy the design acceptance criteria for the shallow angle drop orientation also. From the examination of results presented in Tables 4 through 6, it is observed that the minimum factor of safety is 1.07, which is larger than 1.06 needed for shallow angle drop test.

### 7.1.5 Lead Slump Evaluation

Analysis of the 3-60B cask package under HAC drop test has been performed in the side drop orientation with cask top-end down. Since the top end of the cask has a bolted connection between the lid and the cask body, it is more critical than the bottom-end down orientation which includes no bolted connections. However, the cask is most vulnerable, as far as lead slump is concerned, in the bottom end down orientation. To get a conservative estimate of the lead slump, structural analysis of the cask has been performed with the bottom-end down orientation. The most conservative environmental conditions (cold with no decay heat) have been employed in the analysis. Figure 27 shows the displacement plot during this drop test. The largest relative displacement of 0.3172 in is calculated at the bolting ring-lead interface. It

should be noted this is the total relative displacement. In considering this to be the lead slump, the elastic recovery of the lead and steel has been neglected.

## 7.2 NCT Drop Tests

The distribution of the NCT drop test loading on various components of the cask, under all the drop orientations, have been obtained by linearly proportioning the corresponding loading from the HAC drop tests.

### 7.2.1 End Drop

The following impact limiter reactions are obtained from Reference 4.

Cold Conditions =  $1.338 \times 10^6$  lb (Table 2 and Figure 16 of Reference 4)

Hot Conditions =  $1.103 \times 10^6$  lb (Table 2 and Figure 20 of Reference 4)

Conservatively use the maximum of the two reactions for the analyses of all environmental conditions. The NCT end drop rigid body acceleration is:

$$= 2 \times 1.338 \times 10^6 / 76,200 = 35.12 \text{ g}$$

The acceleration used in the HAC end drop analyses is 150 g. The mechanical loading applied in the HAC end drop analyses may be proportioned with 35.12 g. The ratio of the two loadings is:

$$R_{\text{end}} = 35.12/150 = 0.2341 \quad \gg \text{ For conservatism use } 0.3025$$

#### Impact limiter Inertia

$$p_{\text{I.L.}} = 0.3025 \times 367.3 = 111.11 \text{ psi}$$

#### Payload Inertia

$$p_{\text{lid}} = 0.3025 \times 1,502.5 = 454.5 \text{ psi}$$

#### Cask Body Inertia

$$\text{Cask Body Acceleration} = 0.3025 \times 166 = 50.2 \text{ g}$$

#### Model Analyses

The FEM is analyzed under the above loading for hot (Load Case 1), cold with maximum decay heat (Load Case 2), and cold with no decay heat, environmental conditions. The cask body temperature for Load Cases 1 and 2 are obtained from the hot and cold environmental loading conditions of Reference 10. For Load Case 3 (no internal heat), the entire cask is

assumed to be at -20°F. Figures 28 through 30 show the cask body temperature profile and the pressure distributions used in the FEM for Load Cases 1 through 3.

Figures 31 through 33 show the stress intensity plot in the cask body for the three load cases. Stress intensities are calculated in each major component of the cask and are presented in Tables 10 through 12. These tables also categorize the stresses based on the ASME code and compare them with the corresponding allowable value established in Section 5.0. Factors of safety based on the ratio of allowable stress to the calculated stress are also calculated.

### 7.2.2 Side Drop

The following impact limiter reactions are obtained from Reference 4.

Cold Conditions = 453,400 lb (Table 2 and Figure 24 of Reference 4)

Hot Conditions = 364,800 lb (Table 2 and Figure 28 of Reference 4)

Conservatively use the maximum of the two reactions for the analyses of all environmental conditions. The NCT side drop rigid body acceleration is:

$$= 4 \times 453,400 / 72,400 = 25.05 \text{ g}$$

The acceleration used in the HAC side drop analyses is 120 g. The mechanical loading applied in the HAC side drop analyses may be proportioned with 25.05 g. The ratio of the two loadings is:

$$R_{\text{side}} = 25.05 / 120 = 0.209 \quad \gg \text{ For conservatism use } 0.2354$$

#### Impact Limiter Reactions

Top impact limiter pressure amplitude,

$$p_0 = 0.2354 \times 5,348 = 1,258.9 \text{ psi}$$

Bottom impact limiter pressure amplitude,

$$p_0 = 0.2354 \times 5,078 = 1,195.4 \text{ psi}$$

#### Payload Inertia

Payload inertia pressure amplitude,

$$p_0 = 0.2354 \times 380.5 = 89.57 \text{ psi}$$

#### Cask Body Inertia

$$\text{Cask Body Acceleration} = 0.2354 \times 132.84 = 31.27 \text{ g}$$

Model Analyses

The FEM is analyzed under the above loading for hot (Load Case 1), cold with maximum decay heat (Load Case 2), and cold with no decay heat, environmental conditions. The cask body temperature for Load Cases 1 and 2 are obtained from the hot and cold environmental loading conditions of Reference 10. For Load Case 3 (no internal heat), the entire cask is assumed to be at -20°F. Figures 34 through 36 show the cask body temperature profile and the pressure distributions used in the FEM for Load Cases 1 through 3.

Figures 37 through 39 show the stress intensity plot in the cask body for the three load cases. Stress intensities are calculated in each major component of the cask and are presented in Tables 13 through 15. These tables also categorize the stresses based on the ASME code and compare them with the corresponding allowable value established in Section 5.0. Factors of safety based on the ratio of allowable stress to the calculated stress are also calculated.

**7.2.3 Corner Drop**

The following impact limiter reactions are obtained from Reference 4.

Cold Conditions = 335,300 lb (Table 2 and Figure 32 of Reference 4)

Hot Conditions = 303,208 lb (Table 2 and Figure 36 of Reference 4)

Conservatively use the maximum of the two reactions for the analyses of all environmental conditions. The NCT corner drop rigid body acceleration is:

$$= 2 \times 335,300 / 76,200 = 8.8 \text{ g}$$

The acceleration used in the HAC corner drop analyses is 70 g. The mechanical loading applied in the HAC corner drop analyses may be proportioned with 8.8 g. The ratio of the two loadings is:

$$R_{\text{side}} = 8.8 / 70 = 0.126 \quad \gg \text{ For conservatism use } 0.4292$$

Impact Limiter Reactions

Lid pressure magnitude,

$$p_{\text{lid}} = 0.4292 \times 17,409 = 7,471.9 \text{ psi}$$

Wall pressure amplitude,

$$p_0 = 0.4292 \times 4,769 = 2,046.9 \text{ psi}$$

Impact Limiter Inertia

Baseplate pressure magnitude,

$$p_{i.L} = 0.4292 \times 159.7 = 68.54 \text{ psi}$$

Wall pressure amplitude,

$$p_0 = 0.4292 \times 112.93 = 48.47 \text{ psi}$$

#### Payload Inertia

Lid pressure magnitude

$$p_{\text{lid}} = 0.4292 \times 653.3 = 280.4 \text{ psi}$$

Wall pressure amplitude,

$$p_0 = 0.4292 \times 80.6 = 34.6 \text{ psi}$$

#### Cask Body Inertia

$$\text{Cask Body Axial Acceleration} = 0.4292 \times 72.2 = 30.99 \text{ g}$$

$$\text{Cask Body Lateral Acceleration} = 0.4292 \times 28.14 = 12.1 \text{ g}$$

#### Model Analyses

The FEM is analyzed under the above loading for hot (Load Case 1), cold with maximum decay heat (Load Case 2), and cold with no decay heat, environmental conditions. The cask body temperature for Load Cases 1 and 2 are obtained from the hot and cold environmental loading conditions of Reference 10. For Load Case 3 (no internal heat), the entire cask is assumed to be at -20°F. Figures 40 through 42 show the cask body temperature profile and the pressure distributions used in the FEM for Load Cases 1 through 3.

Figures 43 through 45 show the stress intensity plot in the cask body for the three load cases. Stress intensities are calculated in each major component of the cask and are presented in Tables 16 through 18. These tables also categorize the stresses based on the ASME code and compare them with the corresponding allowable value established in Section 5.0. Factors of safety based on the ratio of allowable stress to the calculated stress are also calculated.

## 8.0 CONCLUSIONS

The results of the analyses performed in this document show that the 3-60B Cask meets the design requirements during all the drop test scenarios specified in 10 CFR 71 code. Therefore, it is concluded that the cask can withstand the drop test requirements during the normal conditions of transport and the hypothetical accident conditions. It is noted that slight deformation of the cask at certain locations is expected during the hypothetical drop tests. However, the components subjected to deformations during these tests are not on the pressure boundary. Therefore, their deformation in no way can prevent the cask from meeting other requirements of the code. A summary of the expected deformation is as follows:

- The skirt of the bolting ring may be subjected to inelastic bending during the side, corner, and shallow angle drop tests (see Figures 17 and 26 for example). This bending will be confined to a small area near the point of impact. The skirt may bend inward at these locations.

## 9.0 ANSYS PRINTOUT AND DATA FILES

The printout of the important data from the program is included with this document in electronic form as Appendix 1. The following is the directory of the data on the DVD.

Volume in drive F is My Disc  
Volume Serial Number is E35A-63CD

Directory of F:\

```

01/04/2008  11:38 AM    <DIR>          1-ft Drop, Corner
12/19/2007  02:52 PM    <DIR>          1-ft Drop, End
01/04/2008  11:15 AM           108,032 1-ft Drop, Result Summary.doc
12/19/2007  02:55 PM    <DIR>          1-ft Drop, Side
01/04/2008  11:44 AM    <DIR>          30-ft Drop, Corner
12/19/2007  02:41 PM    <DIR>          30-ft Drop, End
01/03/2008  04:27 PM           131,584 30-ft Drop, Result Summary.doc
01/03/2008  03:02 PM    <DIR>          30-ft Drop, Side
                2 File(s)          239,616 bytes
                6 Dir(s)              0 bytes free
    
```

Directory of F:\1-ft Drop, Corner

```

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01/04/2008  01:42 PM    <DIR>          ..
10/23/2007  02:10 PM           10,924,071 file.cdb
10/24/2007  05:09 PM           232,390,656 file.rst
10/24/2007  10:18 AM           2,166,410 file.s01
10/24/2007  10:12 AM           2,166,410 file.s02
10/24/2007  10:14 AM           908,444 file.s03
11/20/2007  10:28 AM           97,323 file000.png
11/20/2007  10:30 AM           91,008 file001.png
11/20/2007  10:31 AM           67,746 file002.png
11/20/2007  10:38 AM           153,219 file003.png
11/20/2007  10:39 AM           153,673 file004.png
11/20/2007  10:39 AM           156,392 file005.png
01/03/2008  05:51 PM           3,125 ls1Forging.lis
10/31/2007  10:34 AM           3,125 ls1Inner-Shell.lis
11/01/2007  09:22 AM           3,125 ls1Lid.lis
10/31/2007  10:44 AM           3,125 ls1Outer-Shell.lis
10/29/2007  02:02 PM           3,002,819 ls1post.out
01/03/2008  05:52 PM           3,125 ls2Forging.lis
10/31/2007  10:37 AM           3,125 ls2Inner-Shell.lis
11/01/2007  09:24 AM           3,125 ls2Lid.lis
10/29/2007  02:11 PM           3,002,838 ls2post.out
10/31/2007  12:12 PM           3,125 ls3Baseplates.lis
01/03/2008  05:52 PM           3,125 ls3Forging.lis
10/31/2007  10:38 AM           3,125 ls3Inner-Shell.lis
11/01/2007  09:26 AM           3,125 ls3Lid.lis
    
```

Title Structural Analyses of the 3-60B Cask Under Drop Conditions

Calc. No. ST-504 Rev. 1

Sheet 20 of 27

10/31/2007 10:41 AM 3,125 ls3Outer-Shell.lis  
10/29/2007 02:15 PM 3,002,836 ls3post.out  
12/07/2007 11:14 AM 39,419 Model.txt  
27 File(s) 258,360,764 bytes

Directory of F:\1-ft Drop, End

01/04/2008 01:44 PM <DIR> .  
01/04/2008 01:44 PM <DIR> ..  
10/23/2007 02:09 PM 10,782,335 file.cdb  
10/26/2007 03:35 PM 470,548,480 file.rst  
10/24/2007 10:21 AM 2,025,280 file.s01  
10/26/2007 02:52 PM 2,025,322 file.s02  
10/26/2007 02:50 PM 767,356 file.s03  
11/20/2007 09:08 AM 47,665 file000.png  
11/20/2007 09:07 AM 70,202 file001.png  
11/20/2007 09:09 AM 41,321 file002.png  
11/20/2007 09:23 AM 75,876 file003.png  
11/20/2007 09:26 AM 65,404 file004.png  
11/20/2007 09:28 AM 63,501 file005.png  
10/31/2007 07:33 AM 3,125 ls1Forging.lis  
10/31/2007 07:37 AM 3,125 ls1Outer-shell.lis  
10/29/2007 01:30 PM 3,002,816 ls1post.out  
10/31/2007 08:16 AM 3,125 ls2Forging.lis  
10/31/2007 08:18 AM 3,125 ls2Inner-shell.lis  
10/29/2007 01:34 PM 3,002,834 ls2post.out  
10/31/2007 09:23 AM 3,125 ls3Baseplates.lis  
10/31/2007 08:14 AM 3,125 ls3Forging.lis  
10/31/2007 09:19 AM 3,125 ls3Inner-shell.lis  
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12/07/2007 11:22 AM 39,413 Model.txt  
22 File(s) 495,582,513 bytes

Directory of F:\1-ft Drop, Side

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01/04/2008 01:46 PM <DIR> ..  
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10/26/2007 12:38 PM 234,946,560 file.rst  
10/24/2007 10:30 AM 2,042,171 file.s01  
10/26/2007 11:51 AM 2,042,171 file.s02  
10/26/2007 11:52 AM 784,205 file.s03  
11/20/2007 09:53 AM 137,351 file000.png  
11/20/2007 09:55 AM 121,231 file001.png  
11/20/2007 09:56 AM 93,363 file002.png  
11/20/2007 10:09 AM 154,442 file003.png  
11/20/2007 10:11 AM 138,865 file004.png  
11/20/2007 10:11 AM 141,508 file005.png  
10/31/2007 09:50 AM 3,125 ls1Forging.lis  
10/31/2007 09:29 AM 3,125 ls1Inner-Shell.lis  
10/31/2007 09:32 AM 3,125 ls1Outer-Shell.lis  
10/29/2007 01:45 PM 3,002,817 ls1post.out  
10/31/2007 10:07 AM 3,125 ls2Baseplates.lis  
10/31/2007 09:52 AM 3,125 ls2Forging.lis  
10/31/2007 10:01 AM 3,125 ls2Outer-Shell.lis



```

10/29/2007 01:54 PM          3,002,836 ls2post.out
10/31/2007 10:08 AM           3,125 ls3Baseplates.lis
10/31/2007 09:53 AM           3,125 ls3Forging.lis
10/31/2007 10:05 AM           3,125 ls3Outer-Shell.lis
10/29/2007 01:58 PM          3,002,834 ls3post.out
12/07/2007 11:24 AM           40,131 Model.txt
                24 File(s)    260,551,831 bytes
    
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Directory of F:\30-ft Drop, Corner

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01/04/2008 01:48 PM    <DIR>      ..
10/23/2007 02:10 PM          10,924,071 file.cdb
10/24/2007 06:17 PM          232,980,480 file.rst
10/24/2007 12:31 PM          2,166,410 file.s01
10/24/2007 10:45 AM          2,166,410 file.s02
10/24/2007 10:46 AM           908,444 file.s03
10/23/2007 10:52 AM          1,899,466 file.s04
10/23/2007 01:00 PM          1,499,050 file.s05
11/19/2007 05:37 PM           92,321 file000.png
11/19/2007 05:38 PM          108,580 file001.png
11/19/2007 05:39 PM          101,566 file002.png
11/19/2007 05:40 PM           81,406 file003.png
11/19/2007 05:46 PM          140,328 file004.png
11/19/2007 05:47 PM          143,861 file005.png
11/19/2007 05:47 PM          146,377 file006.png
10/23/2007 07:07 AM          82,903,040 ls1.rst
11/01/2007 02:38 PM           3,125 ls1Forging.lis
11/01/2007 11:58 AM           3,125 ls1Lid.lis
10/29/2007 12:47 PM          3,002,820 ls1post.out
11/01/2007 02:40 PM           3,125 ls2Forging.lis
11/01/2007 12:00 PM           3,125 ls2Lid.lis
10/29/2007 12:53 PM          3,002,839 ls2post.out
11/01/2007 02:40 PM           3,125 ls3Forging.lis
11/01/2007 12:00 PM           3,125 ls3Lid.lis
10/29/2007 12:58 PM          3,002,837 ls3post.out
12/07/2007 11:25 AM           39,421 Model.txt
                25 File(s)    345,328,477 bytes
    
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Directory of F:\30-ft Drop, End

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01/04/2008 01:50 PM    <DIR>      ..
10/23/2007 02:09 PM          10,782,335 file.cdb
11/12/2007 12:13 PM          234,881,024 file.rst
10/24/2007 10:50 AM          2,025,304 file.s01
11/12/2007 11:10 AM          2,025,304 file.s02
10/24/2007 10:55 AM           767,338 file.s03
10/17/2007 07:28 AM           81,233 file000.png
11/02/2007 11:27 AM          152,254 file001.png
10/17/2007 09:12 AM          103,068 file002.png
11/19/2007 02:29 PM           59,387 file003.png
11/19/2007 02:31 PM           69,831 file004.png
11/19/2007 02:48 PM          220,493 file005.png
11/19/2007 02:49 PM           95,780 file006.png
    
```

```

11/19/2007 03:15 PM          135,573 file007.png
11/19/2007 03:40 PM          139,867 file008.png
11/19/2007 03:42 PM          119,828 file009.png
11/19/2007 03:44 PM           89,859 file010.png
11/19/2007 03:53 PM          206,691 file011.png
11/19/2007 03:54 PM          176,300 file012.png
11/19/2007 03:54 PM          173,720 file013.png
11/20/2007 01:18 PM        3,002,775 ls1post.out
10/31/2007 02:25 PM           3,117 ls2Forging.lis
10/31/2007 02:27 PM           3,117 ls2Inner-Shell.lis
11/01/2007 03:16 PM           3,117 ls2Lid.lis
11/20/2007 01:18 PM        3,002,775 ls2post.out
10/31/2007 02:33 PM           3,117 ls3Forging.lis
10/31/2007 02:35 PM           3,117 ls3Inner-Shell.lis
11/01/2007 03:18 PM           3,117 ls3Lid.lis
11/20/2007 01:20 PM        3,002,775 ls3post.out
12/07/2007 11:26 AM          39,714 Model.txt
29 File(s)      261,371,930 bytes
    
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Directory of F:\30-ft Drop, Side

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01/04/2008 01:52 PM    <DIR>      ..
10/23/2007 02:08 PM        10,873,221 file.cdb
10/24/2007 07:00 PM       235,470,848 file.rst
10/24/2007 10:57 AM        2,042,171 file.s01
11/12/2007 11:03 AM        2,042,171 file.s02
10/24/2007 12:07 PM        784,205 file.s03
11/19/2007 04:25 PM        128,170 file000.png
10/31/2007 04:34 PM        114,190 file001.png
11/19/2007 04:26 PM         94,389 file002.png
11/19/2007 04:21 PM        139,130 file003.png
11/19/2007 04:38 PM        163,220 file005.png
11/19/2007 04:39 PM        162,563 file006.png
11/19/2007 04:39 PM        163,381 file007.png
10/31/2007 02:58 PM           3,125 ls1Forging2.lis
10/31/2007 03:03 PM           3,125 ls1Inner-Shell.lis
11/01/2007 03:23 PM           3,125 ls1Lid.lis
10/31/2007 03:07 PM           3,125 ls1Outer-Shell.lis
10/29/2007 12:25 PM       3,002,818 ls1post.out
10/31/2007 03:20 PM           3,125 ls2Baseplates.lis
10/31/2007 02:58 PM           3,125 ls2Forging2.lis
10/31/2007 03:04 PM           3,125 ls2Inner-Shell.lis
11/01/2007 03:24 PM           3,125 ls2Lid.lis
10/31/2007 03:08 PM           3,125 ls2Outer-Shell.lis
10/29/2007 12:29 PM       3,002,837 ls2post.out
10/31/2007 03:22 PM           3,125 ls3Baseplates.lis
10/31/2007 02:59 PM           3,125 ls3Forging2.lis
10/31/2007 03:05 PM           3,125 ls3Inner-Shell.lis
11/01/2007 03:25 PM           3,125 ls3Lid.lis
10/31/2007 03:09 PM           3,125 ls3Outer-Shell.lis
10/29/2007 12:35 PM       3,002,835 ls3post.out
12/07/2007 11:28 AM         40,559 Model.txt
30 File(s)      261,270,458 bytes
    
```

**10.0 APPENDICES**

Appendix 1 Print-out of the ANSYS model data input

Appendix 2 Electronic data on DVD

**Title** Structural Analyses of the 3-60B Cask Under Drop Conditions

**Calc. No.** ST-504 **Rev.** 1

**Sheet** 24 **of** 27

**Tables**

(18 Pages)

**Table 1**

Stress Intensities in 3-60B Cask under 30-ft End Drop – Hot Condition

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	P <sub>m</sub>	59,760	32,855	1.82
	P <sub>m</sub> + P <sub>b</sub>	87,000	32,855	2.65
	F	174,000	32,855	5.30
Inner Shell	P <sub>m</sub>	59,760	17,652	3.39
	P <sub>m</sub> + P <sub>b</sub>	87,000	17,652	4.93
	F	174,000	17,652	9.86
Outer Shell	P <sub>m</sub>	40,080	31,224	1.28
	P <sub>m</sub> + P <sub>b</sub>	60,120	31,224	1.93
	F	140,000	31,224	4.48
Lid	P <sub>m</sub>	40,080	30,311	1.32
	P <sub>m</sub> + P <sub>b</sub>	60,120	30,311	1.98
	F	140,000	30,311	4.62
Base Plates	P <sub>m</sub>	40,080	14,924	2.69
	P <sub>m</sub> + P <sub>b</sub>	60,120	14,924	4.03
	F	140,000	14,924	9.38
Seal Plates	P <sub>m</sub> + P <sub>b</sub>	60,120	4,185 <sup>(3)</sup>	14.37
Bolts	P <sub>m</sub>	105,000	9,023	11.64
	P <sub>m</sub> + P <sub>b</sub>	150,000	9,023	16.62
	F	300,000	9,023	33.25

Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as P<sub>m</sub> and P<sub>m</sub> + P<sub>b</sub> stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) The maximum stress intensity in the seal plates is 51,854 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).

**Table 2**

Stress Intensities in 3-60B Cask under 30-ft End Drop – Cold Condition (Max. Decay Heat)

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	P <sub>m</sub>	59,760	50,491	1.18
	P <sub>m</sub> + P <sub>b</sub>	87,000	50,491	1.72
	F	174,000	50,491	3.45
Inner Shell	P <sub>m</sub>	59,760	38,207	1.56
	P <sub>m</sub> + P <sub>b</sub>	87,000	38,207	2.28
	F	174,000	38,207	4.55
Outer Shell	P <sub>m</sub>	40,080	24,782	1.62
	P <sub>m</sub> + P <sub>b</sub>	60,120	24,782	2.43
	F	140,000	24,782	5.65
Lid	P <sub>m</sub>	40,080	33,945	1.18
	P <sub>m</sub> + P <sub>b</sub>	60,120	33,945	1.77
	F	140,000	33,945	4.12
Base Plates	P <sub>m</sub>	40,080	24,661	1.63
	P <sub>m</sub> + P <sub>b</sub>	60,120	24,661	2.44
	F	140,000	24,661	5.68
Seal Plates	P <sub>m</sub> + P <sub>b</sub>	60,120	4,187 <sup>(3)</sup>	14.36
Bolts	P <sub>m</sub>	105,000	7,592	13.83
	P <sub>m</sub> + P <sub>b</sub>	150,000	7,592	19.76
	F	300,000	7,592	39.52

## Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as P<sub>m</sub> and P<sub>m</sub> + P<sub>b</sub> stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) The maximum stress intensity in the seal plates is 56,497 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).

**Table 3**

Stress Intensities in 3-60B Cask under 30-ft End Drop – Cold Condition (No Decay Heat)

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	P <sub>m</sub>	59,760	44,400	1.35
	P <sub>m</sub> + P <sub>b</sub>	87,000	60,610	1.44
	F	174,000	58,779	2.96
Inner Shell	P <sub>m</sub>	59,760	43,130	1.39
	P <sub>m</sub> + P <sub>b</sub>	87,000	46,060	1.89
	F	174,000	43,700	3.98
Outer Shell	P <sub>m</sub>	40,080	24,687	1.62
	P <sub>m</sub> + P <sub>b</sub>	60,120	24,687	2.44
	F	140,000	24,687	5.67
Lid	P <sub>m</sub>	40,080	5,482	7.31
	P <sub>m</sub> + P <sub>b</sub>	60,120	35,126	1.71
	F	140,000	35,126	3.99
Base Plates	P <sub>m</sub>	40,080	27,593	1.45
	P <sub>m</sub> + P <sub>b</sub>	60,120	27,593	2.18
	F	140,000	27,593	5.07
Seal Plates	P <sub>m</sub> + P <sub>b</sub>	60,120	4,971 <sup>(3)</sup>	12.09
Bolts	P <sub>m</sub>	105,000	7,442	14.11
	P <sub>m</sub> + P <sub>b</sub>	150,000	7,442	20.16
	F	300,000	7,442	40.31

## Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as P<sub>m</sub> and P<sub>m</sub> + P<sub>b</sub> stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) The maximum stress intensity in the seal plates is 57,706 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).

**Table 4**

Stress Intensities in 3-60B Cask under 30-ft Side Drop – Hot Condition

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	P <sub>m</sub>	59,760	45,960 <sup>(3)</sup>	1.30
	P <sub>m</sub> + P <sub>b</sub>	87,000	81,730 <sup>(3)</sup>	1.07
	F	174,000	125,360	1.39
Inner Shell	P <sub>m</sub>	59,760	36,420	1.64
	P <sub>m</sub> + P <sub>b</sub>	87,000	44,210	1.97
	F	174,000	44,216	3.94
Outer Shell	P <sub>m</sub>	40,080	33,800	1.19
	P <sub>m</sub> + P <sub>b</sub>	60,120	44,150	1.36
	F	140,000	44,151	3.17
Lid	P <sub>m</sub>	40,080	26,280 <sup>(4)</sup>	1.53
	P <sub>m</sub> + P <sub>b</sub>	60,120	32,940 <sup>(6)</sup>	1.83
	F	140,000	40,684	3.44
Base Plates	P <sub>m</sub>	40,080	31,876	1.26
	P <sub>m</sub> + P <sub>b</sub>	60,120	31,876	1.89
	F	140,000	31,876	4.39
Seal Plates	P <sub>m</sub> + P <sub>b</sub>	60,120	45,515 <sup>(5)</sup>	1.32
Bolts	P <sub>m</sub>	105,000	57,103	1.84
	P <sub>m</sub> + P <sub>b</sub>	150,000	57,103	2.63
	F	300,000	57,103	5.25

Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as P<sub>m</sub> and P<sub>m</sub> + P<sub>b</sub> stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) Stress intensity in parts other than the skirt. The skirt is expected to deform under HAC side drop.
- (4) Average value over the section of maximum stress intensity.
- (5) The maximum stress intensity in the seal plates is 104,460 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).
- (6) The reported stress here is the maximum principle stress (tensile).



**Table 5**

Stress Intensities in 3-60B Cask under 30-ft Side Drop – Cold Condition (Max. Decay Heat)

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	P <sub>m</sub>	59,760	45,210 <sup>(3)</sup>	1.32
	P <sub>m</sub> + P <sub>b</sub>	87,000	81,460 <sup>(3)</sup>	1.07
	F	174,000	127,328	1.37
Inner Shell	P <sub>m</sub>	59,760	37,560	1.59
	P <sub>m</sub> + P <sub>b</sub>	87,000	42,230	2.06
	F	174,000	42,230	4.12
Outer Shell	P <sub>m</sub>	40,080	35,230	1.14
	P <sub>m</sub> + P <sub>b</sub>	60,120	47,480	1.27
	F	140,000	47,487	2.95
Lid	P <sub>m</sub>	40,080	27,640 <sup>(4)</sup>	1.45
	P <sub>m</sub> + P <sub>b</sub>	60,120	35,884 <sup>(6)</sup>	1.68
	F	140,000	42,435	3.30
Base Plates	P <sub>m</sub>	40,080	26,210	1.53
	P <sub>m</sub> + P <sub>b</sub>	60,120	50,500	1.19
	F	140,000	52,020	2.69
Seal Plates	P <sub>m</sub> + P <sub>b</sub>	60,120	46,227 <sup>(5)</sup>	1.30
Bolts	P <sub>m</sub>	105,000	55,860	1.88
	P <sub>m</sub> + P <sub>b</sub>	150,000	55,860	2.69
	F	300,000	55,860	5.37

Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as P<sub>m</sub> and P<sub>m</sub> + P<sub>b</sub> stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) Stress intensity in parts other than the skirt. The skirt is expected to deform under HAC side drop.
- (4) Average value over the section of maximum stress intensity.
- (5) The maximum stress intensity in the seal plates is 106,330 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).
- (6) The reported stress here is the maximum principle stress (tensile).

**Table 6**

Stress Intensities in 3-60B Cask under 30-ft Side Drop – Cold Condition (No Decay Heat)

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	P <sub>m</sub>	59,760	44,900 <sup>(3)</sup>	1.33
	P <sub>m</sub> + P <sub>b</sub>	87,000	81,360 <sup>(3)</sup>	1.07
	F	174,000	128,091	1.36
Inner Shell	P <sub>m</sub>	59,760	40,200	1.49
	P <sub>m</sub> + P <sub>b</sub>	87,000	43,390	2.01
	F	174,000	43,486	4.00
Outer Shell	P <sub>m</sub>	40,080	36,710	1.09
	P <sub>m</sub> + P <sub>b</sub>	60,120	49,360	1.22
	F	140,000	49,364	2.84
Lid	P <sub>m</sub>	40,080	27,360 <sup>(4)</sup>	1.46
	P <sub>m</sub> + P <sub>b</sub>	60,120	35,719 <sup>(6)</sup>	1.68
	F	140,000	41,878	3.34
Base Plates	P <sub>m</sub>	40,080	29,690	1.35
	P <sub>m</sub> + P <sub>b</sub>	60,120	53,950	1.11
	F	140,000	57,405	2.44
Seal Plates	P <sub>m</sub> + P <sub>b</sub>	60,120	45,153 <sup>(5)</sup>	1.33
Bolts	P <sub>m</sub>	105,000	54,432	1.93
	P <sub>m</sub> + P <sub>b</sub>	150,000	54,432	2.76
	F	300,000	54,432	5.51

Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as P<sub>m</sub> and P<sub>m</sub> + P<sub>b</sub> stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) Stress intensity in parts other than the skirt. The skirt is expected to deform under HAC side drop.
- (4) Average value over the section of maximum stress intensity.
- (5) The maximum stress intensity in the seal plates is 103,850 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).
- (6) The reported stress here is the maximum principle stress (tensile).

**Table 7****Stress Intensities in 3-60B Cask under 30-ft Corner Drop – Hot Condition**

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	P <sub>m</sub>	59,760	41,620	1.44
	P <sub>m</sub> + P <sub>b</sub>	87,000	50,330	1.73
	F	174,000	139,619	1.25
Inner Shell	P <sub>m</sub>	59,760	35,571	1.68
	P <sub>m</sub> + P <sub>b</sub>	87,000	35,571	2.45
	F	174,000	35,571	4.89
Outer Shell	P <sub>m</sub>	40,080	31,297	1.28
	P <sub>m</sub> + P <sub>b</sub>	60,120	31,297	1.92
	F	140,000	31,297	4.47
Lid	P <sub>m</sub>	40,080	27,550	1.45
	P <sub>m</sub> + P <sub>b</sub>	60,120	42,817 <sup>(4)</sup>	1.40
	F	140,000	100,030	1.40
Base Plates	P <sub>m</sub>	40,080	10,203	3.93
	P <sub>m</sub> + P <sub>b</sub>	60,120	10,203	5.89
	F	140,000	10,203	13.72
Seal Plates	P <sub>m</sub> + P <sub>b</sub>	60,120	34,765 <sup>(3)</sup>	1.73
Bolts	P <sub>m</sub>	105,000	27,642	3.80
	P <sub>m</sub> + P <sub>b</sub>	150,000	27,642	5.43
	F	300,000	27,642	10.85

## Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as P<sub>m</sub> and P<sub>m</sub> + P<sub>b</sub> stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) The maximum stress intensity in the seal plates is 185,160 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).
- (4) The reported stress here is the maximum principle stress (tensile).

**Table 8**Stress Intensities in 3-60B Cask under 30-ft Corner Drop – Cold Condition (Max. Decay Heat)

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	$P_m$	59,760	44,450	1.34
	$P_m + P_b$	87,000	47,750	1.82
	F	174,000	130,470	1.33
Inner Shell	$P_m$	59,760	50,708	1.18
	$P_m + P_b$	87,000	50,708	1.72
	F	174,000	50,708	3.43
Outer Shell	$P_m$	40,080	25,953	1.54
	$P_m + P_b$	60,120	25,953	2.32
	F	140,000	25,953	5.39
Lid	$P_m$	40,080	26,240	1.53
	$P_m + P_b$	60,120	42,737 <sup>(4)</sup>	1.41
	F	140,000	96,158	1.46
Base Plates	$P_m$	40,080	16,204	2.47
	$P_m + P_b$	60,120	16,204	3.71
	F	140,000	16,204	8.64
Seal Plates	$P_m + P_b$	60,120	37,369 <sup>(3)</sup>	1.61
Bolts	$P_m$	105,000	25,437	4.13
	$P_m + P_b$	150,000	25,437	5.90
	F	300,000	25,437	11.79

## Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as  $P_m$  and  $P_m + P_b$  stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) The maximum stress intensity in the seal plates is 173,420 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).
- (4) The reported stress here is the maximum principle stress (tensile).

**Table 9**

Stress Intensities in 3-60B Cask under 30-ft Corner Drop – Cold Condition (No Decay Heat)

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	P <sub>m</sub>	59,760	46,380	1.29
	P <sub>m</sub> + P <sub>b</sub>	87,000	52,140	1.67
	F	174,000	126,480	1.38
Inner Shell	P <sub>m</sub>	59,760	55,586	1.08
	P <sub>m</sub> + P <sub>b</sub>	87,000	55,586	1.57
	F	174,000	55,586	3.13
Outer Shell	P <sub>m</sub>	40,080	26,917	1.49
	P <sub>m</sub> + P <sub>b</sub>	60,120	26,917	2.23
	F	140,000	26,917	5.20
Lid	P <sub>m</sub>	40,080	26,050	1.54
	P <sub>m</sub> + P <sub>b</sub>	60,120	42,578 <sup>(4)</sup>	1.41
	F	140,000	95,863	1.46
Base Plates	P <sub>m</sub>	40,080	21,989	1.82
	P <sub>m</sub> + P <sub>b</sub>	60,120	21,989	2.73
	F	140,000	21,989	6.37
Seal Plates	P <sub>m</sub> + P <sub>b</sub>	60,120	37,834 <sup>(3)</sup>	1.59
Bolts	P <sub>m</sub>	105,000	26,079	4.03
	P <sub>m</sub> + P <sub>b</sub>	150,000	26,079	5.75
	F	300,000	26,079	11.50

Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as P<sub>m</sub> and P<sub>m</sub> + P<sub>b</sub> stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) The maximum stress intensity in the seal plates is 169,950 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).
- (4) The reported stress here is the maximum principle stress (tensile).

**Table 10**

Stress Intensities in 3-60B Cask under 1-ft End Drop – Hot Condition

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	$P_m$	24,900	8,192	3.04
	$P_m + P_b$	37,350	15,310	2.44
	F	74,700	15,342	4.87
Inner Shell	$P_m$	24,900	4,245	5.87
	$P_m + P_b$	37,350	4,245	8.80
	F	74,700	4,245	17.60
Outer Shell	$P_m$	16,700	13,760	1.21
	$P_m + P_b$	25,050	15,030	1.67
	F	50,100	15,035	3.33
Lid	$P_m$	16,700	10,138	1.65
	$P_m + P_b$	25,050	10,138	2.47
	F	50,100	10,138	4.94
Base Plates	$P_m$	16,700	10,182	1.64
	$P_m + P_b$	25,050	10,182	2.46
	F	50,100	10,182	4.92
Seal Plates	$P_m + P_b$	25,050	16,808	1.49
Bolts	$P_m$	60,000	6,725	8.92
	$P_m + P_b$	90,000	6,725	13.38
	F	150,000	6,725	22.30

Notes:

(1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as  $P_m$  and  $P_m + P_b$  stress intensities.

(2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)

**Table 11**

Stress Intensities in 3-60B Cask under 1-ft End Drop – Cold Condition (Max. Decay Heat)

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	$P_m$	24,900	16,850	1.48
	$P_m + P_b$	37,350	22,960	1.63
	F	74,700	22,965	3.25
Inner Shell	$P_m$	24,900	16,310	1.53
	$P_m + P_b$	37,350	17,040	2.19
	F	74,700	17,043	4.38
Outer Shell	$P_m$	16,700	7,562	2.21
	$P_m + P_b$	25,050	7,562	3.31
	F	50,100	7,562	6.63
Lid	$P_m$	16,700	10,320	1.62
	$P_m + P_b$	25,050	10,320	2.43
	F	50,100	10,320	4.85
Base Plates	$P_m$	16,700	12,590	1.33
	$P_m + P_b$	25,050	12,590	1.99
	F	50,100	12,590	3.98
Seal Plates	$P_m + P_b$	25,050	17,356	1.44
Bolts	$P_m$	60,000	3,646	16.46
	$P_m + P_b$	90,000	3,646	24.68
	F	150,000	3,646	41.14

Notes:

(1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as  $P_m$  and  $P_m + P_b$  stress intensities.

(2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)

**Table 12**

Stress Intensities in 3-60B Cask under 1-ft End Drop – Cold Condition (No Decay Heat)

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	$P_m$	24,900	20,970	1.19
	$P_m + P_b$	37,350	28,920	1.29
	F	74,700	28,928	2.58
Inner Shell	$P_m$	24,900	20,190	1.23
	$P_m + P_b$	37,350	21,180	1.76
	F	74,700	21,183	3.53
Outer Shell	$P_m$	16,700	7,467	2.24
	$P_m + P_b$	25,050	7,467	3.35
	F	50,100	7,467	6.71
Lid	$P_m$	16,700	11,125	1.50
	$P_m + P_b$	25,050	11,125	2.25
	F	50,100	11,125	4.50
Base Plates	$P_m$	16,700	10,210	1.64
	$P_m + P_b$	25,050	17,040	1.47
	F	50,100	18,208	2.75
Seal Plates	$P_m + P_b$	25,050	19,186	1.31
Bolts	$P_m$	60,000	5,301	11.32
	$P_m + P_b$	90,000	5,301	16.98
	F	150,000	5,301	28.30

## Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as  $P_m$  and  $P_m + P_b$  stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)



**Table 13**

Stress Intensities in 3-60B Cask under 1-ft Side Drop – Hot Condition

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	P <sub>m</sub>	24,900	18,480	1.35
	P <sub>m</sub> + P <sub>b</sub>	37,350	30,410	1.23
	F	74,700	30,441	2.45
Inner Shell	P <sub>m</sub>	24,900	14,490	1.72
	P <sub>m</sub> + P <sub>b</sub>	37,350	16,470	2.27
	F	74,700	16,467	4.54
Outer Shell	P <sub>m</sub>	16,700	9,915	1.68
	P <sub>m</sub> + P <sub>b</sub>	25,050	20,060	1.25
	F	50,100	20,069	2.50
Lid	P <sub>m</sub>	16,700	7,440	2.24
	P <sub>m</sub> + P <sub>b</sub>	25,050	7,440	3.37
	F	50,100	7,440	6.73
Base Plates	P <sub>m</sub>	16,700	12,645	1.32
	P <sub>m</sub> + P <sub>b</sub>	25,050	12,645	1.98
	F	50,100	12,645	3.96
Seal Plates	P <sub>m</sub> + P <sub>b</sub>	25,050	5,415 <sup>(3)</sup>	4.63
Bolts	P <sub>m</sub>	60,000	24,328	2.47
	P <sub>m</sub> + P <sub>b</sub>	90,000	24,328	3.70
	F	150,000	24,328	6.17

Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as P<sub>m</sub> and P<sub>m</sub> + P<sub>b</sub> stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) The maximum stress intensity in the seal plates is 22,040 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).

**Table 14**

Stress Intensities in 3-60B Cask under 1-ft Side Drop – Cold Condition (Max. Decay Heat)

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	P <sub>m</sub>	24,900	19,060	1.31
	P <sub>m</sub> + P <sub>b</sub>	37,350	31,240	1.20
	F	74,700	31,247	2.39
Inner Shell	P <sub>m</sub>	24,900	13,051	1.91
	P <sub>m</sub> + P <sub>b</sub>	37,350	13,051	2.86
	F	74,700	13,051	5.72
Outer Shell	P <sub>m</sub>	16,700	11,240	1.49
	P <sub>m</sub> + P <sub>b</sub>	25,050	14,810	1.69
	F	50,100	14,816	3.38
Lid	P <sub>m</sub>	16,700	10,147	1.65
	P <sub>m</sub> + P <sub>b</sub>	25,050	10,147	2.47
	F	50,100	10,147	4.94
Base Plates	P <sub>m</sub>	16,700	10,280	1.62
	P <sub>m</sub> + P <sub>b</sub>	25,050	16,960	1.48
	F	50,100	18,373	2.73
Seal Plates	P <sub>m</sub> + P <sub>b</sub>	25,050	9,815 <sup>(3)</sup>	2.55
Bolts	P <sub>m</sub>	60,000	21,543	2.79
	P <sub>m</sub> + P <sub>b</sub>	90,000	21,543	4.18
	F	150,000	21,543	6.96

Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as P<sub>m</sub> and P<sub>m</sub> + P<sub>b</sub> stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) The maximum stress intensity in the seal plates is 26,446 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).

**Table 15**

Stress Intensities in 3-60B Cask under 1-ft Side Drop – Cold Condition (No Decay Heat)

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	P <sub>m</sub>	24,900	19,340	1.29
	P <sub>m</sub> + P <sub>b</sub>	37,350	31,690	1.18
	F	74,700	31,694	2.36
Inner Shell	P <sub>m</sub>	24,900	16,167	1.54
	P <sub>m</sub> + P <sub>b</sub>	37,350	16,167	2.31
	F	74,700	16,167	4.62
Outer Shell	P <sub>m</sub>	16,700	12,440	1.34
	P <sub>m</sub> + P <sub>b</sub>	25,050	16,800	1.49
	F	50,100	16,807	2.98
Lid	P <sub>m</sub>	16,700	11,179	1.49
	P <sub>m</sub> + P <sub>b</sub>	25,050	11,179	2.24
	F	50,100	11,179	4.48
Base Plates	P <sub>m</sub>	16,700	14,290	1.17
	P <sub>m</sub> + P <sub>b</sub>	25,050	22,330	1.12
	F	50,100	24,154	2.07
Seal Plates	P <sub>m</sub> + P <sub>b</sub>	25,050	10,399 <sup>(3)</sup>	2.41
Bolts	P <sub>m</sub>	60,000	19,916	3.01
	P <sub>m</sub> + P <sub>b</sub>	90,000	19,916	4.52
	F	150,000	19,916	7.53

Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as P<sub>m</sub> and P<sub>m</sub> + P<sub>b</sub> stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) The maximum stress intensity in the seal plates is 24,543 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).

**Table 16**

Stress Intensities in 3-60B Cask under 1-ft Corner Drop – Hot Condition

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	$P_m$	24,900	19,580 <sup>(6)</sup>	1.27
	$P_m + P_b$	37,350	26,170 <sup>(6)</sup>	1.43
	F	74,700	55,516	1.35
Inner Shell <sup>(3)</sup>	$P_L$	24,900	13,350	1.87
	$P_L + P_b$	37,350	14,530	2.57
	F	74,700	14,534	5.14
Outer Shell	$P_m$	16,700	8,248	2.02
	$P_m + P_b$	25,050	16,270	1.54
	F	50,100	16,269	3.08
Lid	$P_m$	16,700	9,966	1.68
	$P_m + P_b$	25,050	18,347 <sup>(5)</sup>	1.37
	F	50,100	41,359	1.21
Base Plates	$P_m$	16,700	10,896	1.53
	$P_m + P_b$	25,050	10,896	2.30
	F	50,100	10,896	4.60
Seal Plates	$P_m + P_b$	25,050	12,606 <sup>(4)</sup>	1.99
Bolts	$P_m$	60,000	18,243	3.29
	$P_m + P_b$	90,000	18,243	4.93
	F	150,000	18,243	8.22

Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as  $P_m$  and  $P_m + P_b$  stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) The stresses in the inner shell under corner drop loading are mostly longitudinal. These stresses are the highest near the impact location and subside greatly away from the plane of impact. Therefore, they are classified as average linearized stress,  $P_L$  and not  $P_m$ .
- (4) The maximum stress intensity in the seal plates is 77,292 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).
- (5) The reported stress here is the maximum principle stress (tensile).
- (6) Membrane and membrane plus bending stresses calculated in non-skirt elements.

**Table 17**

Stress Intensities in 3-60B Cask under 1-ft Corner Drop – Cold Condition (Max. Decay Heat)

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	P <sub>m</sub>	24,900	21,730 <sup>(6)</sup>	1.15
	P <sub>m</sub> + P <sub>b</sub>	37,350	25,680 <sup>(6)</sup>	1.45
	F	74,700	52,478	1.42
Inner Shell <sup>(3)</sup>	P <sub>L</sub>	37,350	26,480	1.41
	P <sub>L</sub> + P <sub>b</sub>	37,350	27,570	1.35
	F	74,700	27,569	2.71
Outer Shell	P <sub>m</sub>	16,700	12,611	1.32
	P <sub>m</sub> + P <sub>b</sub>	25,050	12,611	1.99
	F	50,100	12,611	3.97
Lid	P <sub>m</sub>	16,700	9,943	1.68
	P <sub>m</sub> + P <sub>b</sub>	25,050	18,344 <sup>(5)</sup>	1.37
	F	50,100	39,239	1.28
Base Plates	P <sub>m</sub>	16,700	11,656	1.43
	P <sub>m</sub> + P <sub>b</sub>	25,050	11,656	2.15
	F	50,100	11,656	4.30
Seal Plates	P <sub>m</sub> + P <sub>b</sub>	25,050	18,934 <sup>(4)</sup>	1.32
Bolts	P <sub>m</sub>	60,000	14,026	4.28
	P <sub>m</sub> + P <sub>b</sub>	90,000	14,026	6.42
	F	150,000	14,026	10.69

Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as P<sub>m</sub> and P<sub>m</sub> + P<sub>b</sub> stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) The stresses in the inner shell under corner drop loading are mostly longitudinal. These stresses are the highest near the impact location and subside greatly away from the plane of impact. Therefore, they are classified as average linearized stress, P<sub>L</sub> and not P<sub>m</sub>.
- (4) The maximum stress intensity in the seal plates is 71,591 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).
- (5) The reported stress here is the maximum principle stress (tensile).
- (6) Membrane and membrane plus bending stresses calculated in non-skirt elements.

**Table 18**

Stress Intensities in 3-60B Cask under 1-ft Corner Drop – Cold Condition (No Decay Heat)

Component	Stress Category	Allowable S.I. (psi)	Calculated S.I. <sup>(1)</sup> (psi)	F.S. <sup>(2)</sup>
Bolting Ring	$P_m$	24,900	23,580 <sup>(6)</sup>	1.06
	$P_m + P_b$	37,350	30,260 <sup>(6)</sup>	1.23
	F	74,700	49,660	1.50
Inner Shell <sup>(3)</sup>	$P_L$	37,350	30,150	1.24
	$P_L + P_b$	37,350	32,220	1.20
	F	74,700	32,217	2.32
Outer Shell	$P_m$	16,700	9,999	1.67
	$P_m + P_b$	25,050	14,390	1.74
	F	50,100	14,387	3.48
Lid	$P_m$	16,700	9,940	1.68
	$P_m + P_b$	25,050	18,199 <sup>(5)</sup>	1.38
	F	50,100	39,298	1.27
Base Plates	$P_m$	16,700	10,880	1.53
	$P_m + P_b$	25,050	18,310	1.37
	F	50,100	18,310	2.74
Seal Plates	$P_m + P_b$	25,050	19,456 <sup>(4)</sup>	1.29
Bolts	$P_m$	60,000	13,725	4.37
	$P_m + P_b$	90,000	13,725	6.56
	F	150,000	13,725	10.93

## Notes:

- (1) Unless otherwise indicated in this column, the peak stress intensity (F) values have been conservatively reported as  $P_m$  and  $P_m + P_b$  stress intensities.
- (2) Factor of Safety, F.S. = (Allowable S.I.) / (Calculated S.I.)
- (3) The stresses in the inner shell under corner drop loading are mostly longitudinal. These stresses are the highest near the impact location and subside greatly away from the plane of impact. Therefore, they are classified as average linearized stress,  $P_L$  and not  $P_m$ .
- (4) The maximum stress intensity in the seal plates is 69,165 psi. However, the plates are under compression and the maximum stress intensity may be categorized as bearing stress. The reported stress here is the maximum principal stress (tensile).
- (5) The reported stress here is the maximum principle stress (tensile).
- (6) Membrane and membrane plus bending stresses calculated in non-skirt elements.

Title Structural Analyses of the 3-60B Cask Under Drop Conditions

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Sheet 25 of 27

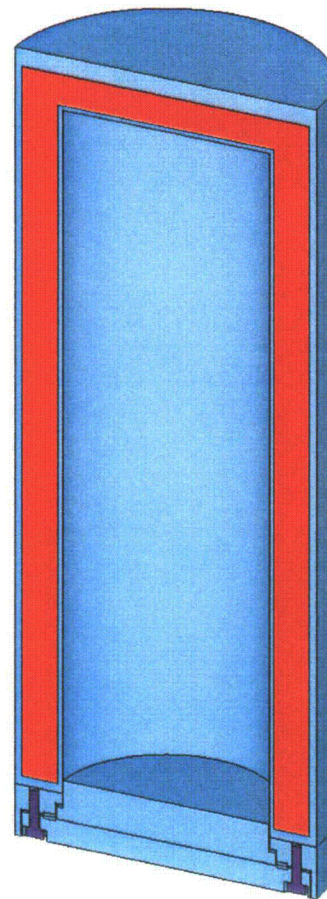
**Figures**




(45 Pages)

ELEMENTS  
MAT NUM

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LEGEND	
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	Carbon Steel
	Lead

Title \_\_\_\_\_ Structural Analyses of the 3-60B Cask Under Drop Conditions  
Calc. No. ST-504 (Figures) \_\_\_\_\_ Rev. 1 \_\_\_\_\_ Sheet 1 of 45

Figure 1

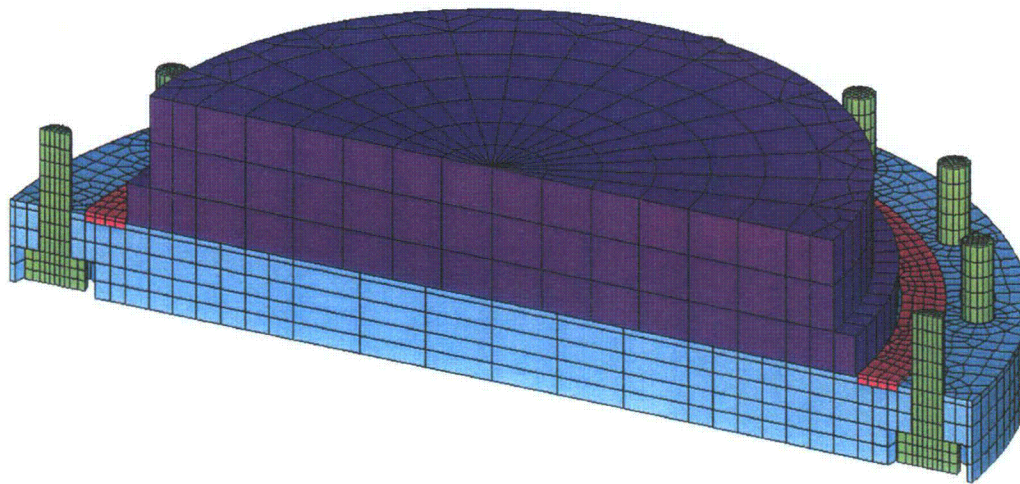
Finite Element Model of the 3-60B Cask Identifying the Cask Components with Material Numbers



ELEMENTS  
REAL NUM

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PLOT NO. 1



Title \_\_\_\_\_ Structural Analyses of the 3-60B Cask Under Drop Conditions  
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Figure 2  
Finite Element Model of the Lid, Seal Plate and Bolts

ELEMENTS  
REAL NUM

ANSYS

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PLOT NO. 1

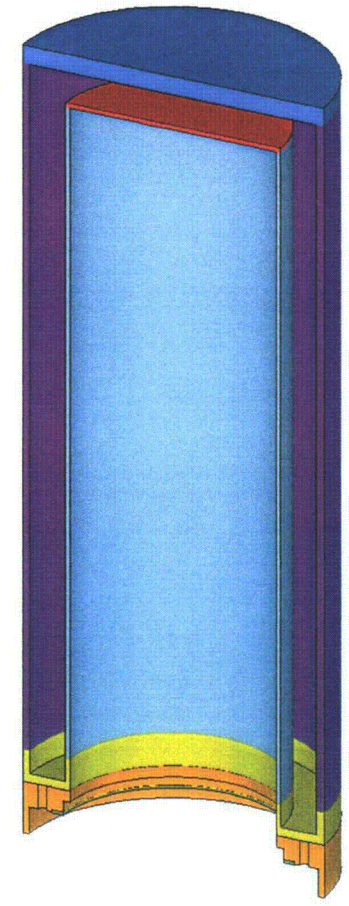
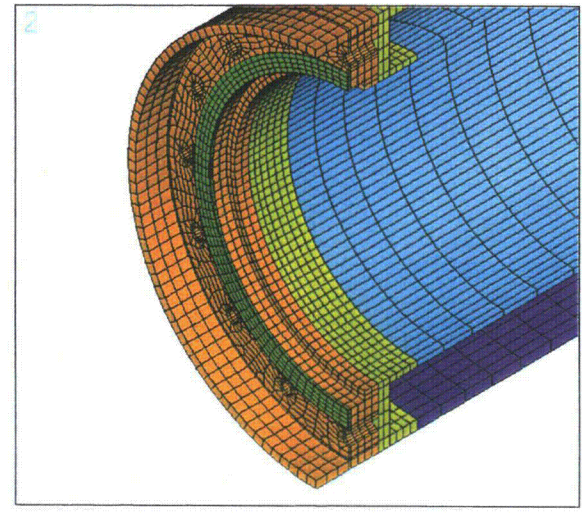
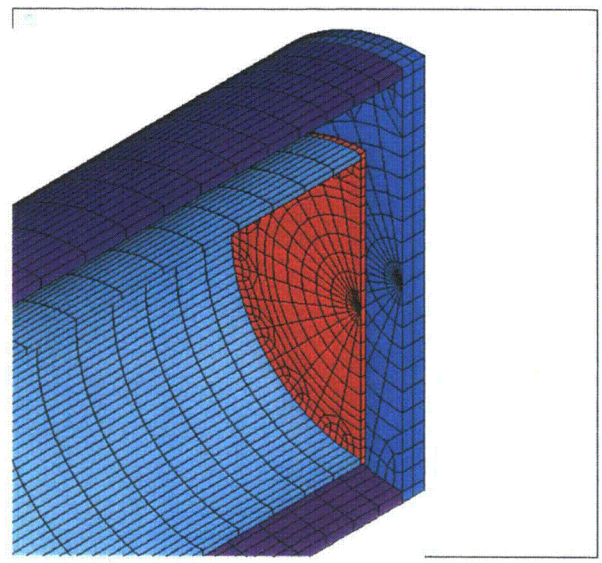
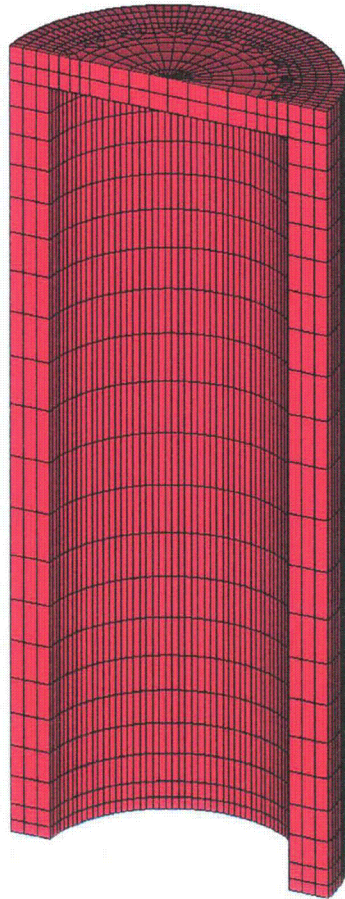


Figure 3  
Finite Element Model of the cask Body without the Lead

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ELEMENTS  
REAL NUM

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Figure 4  
Finite Element Model of the Lead



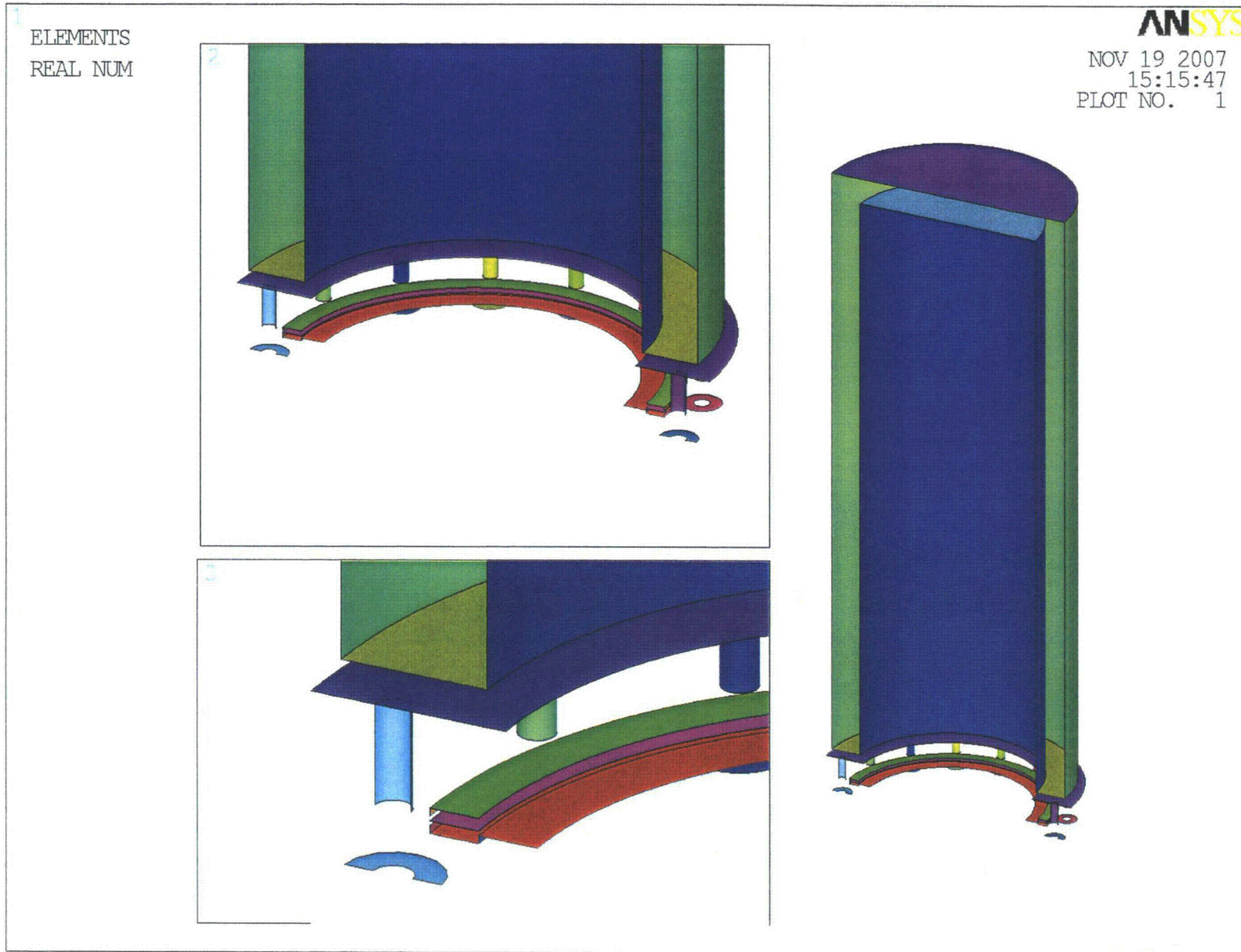
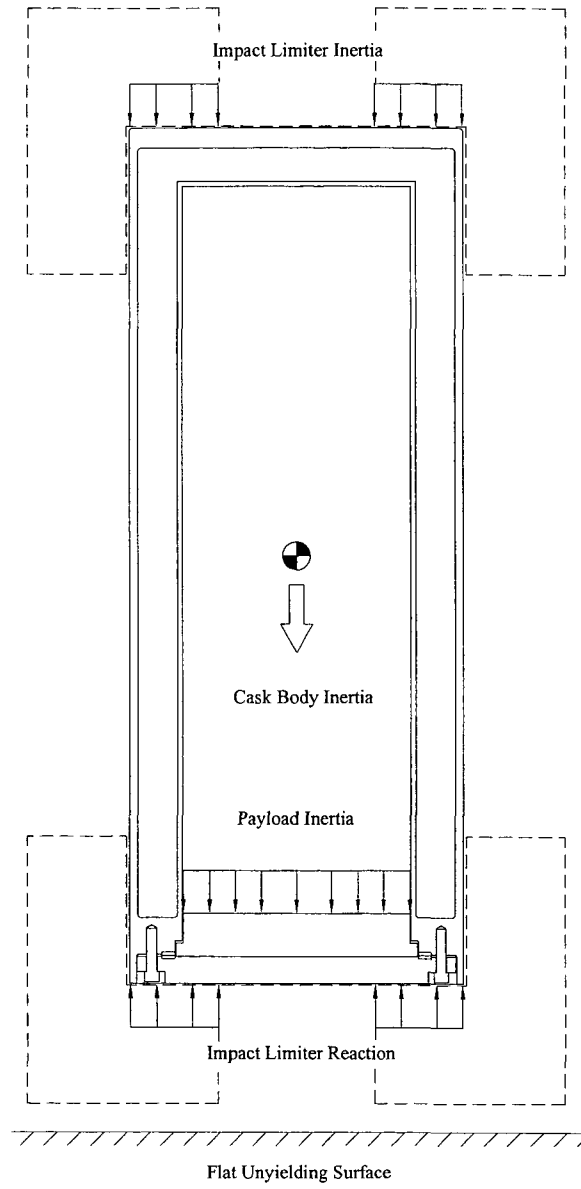


Figure 5

Finite Element Model of the Contact-Target Elements (Only Target Elements Shown)



**Figure 6**  
**Load Distribution on the Model during End Drop**

ELEMENTS

PRES-NORM

TEMPERATURES

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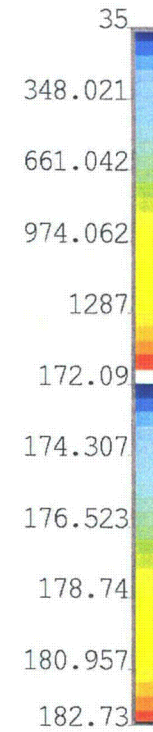
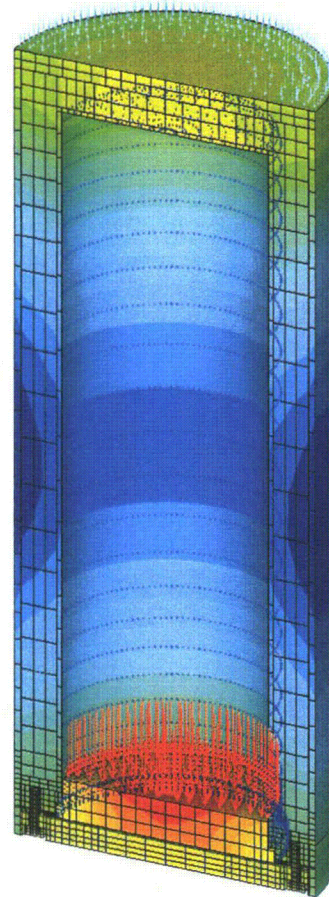
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PLOT NO. 1



30-ft End Drop - Hot Condition

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Figure 7

Temperature Profile and Pressure Distribution Used for 30-ft End Drop – Load Combination No.1



ELEMENTS

PRES-NORM

TEMPERATURES

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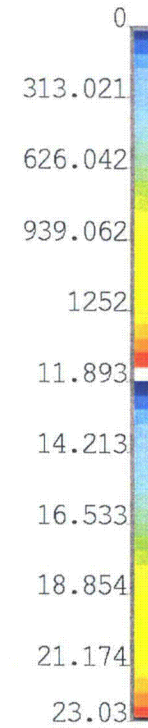
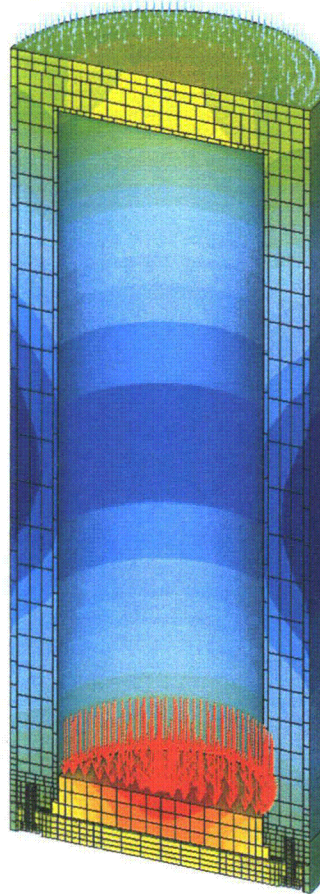
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PLOT NO. 1



30-ft End Drop - Cold Condition (Max. Decay Heat)

Figure 8

Temperature Profile and Pressure Distribution Used for 30-ft End Drop - Load Combination No.2



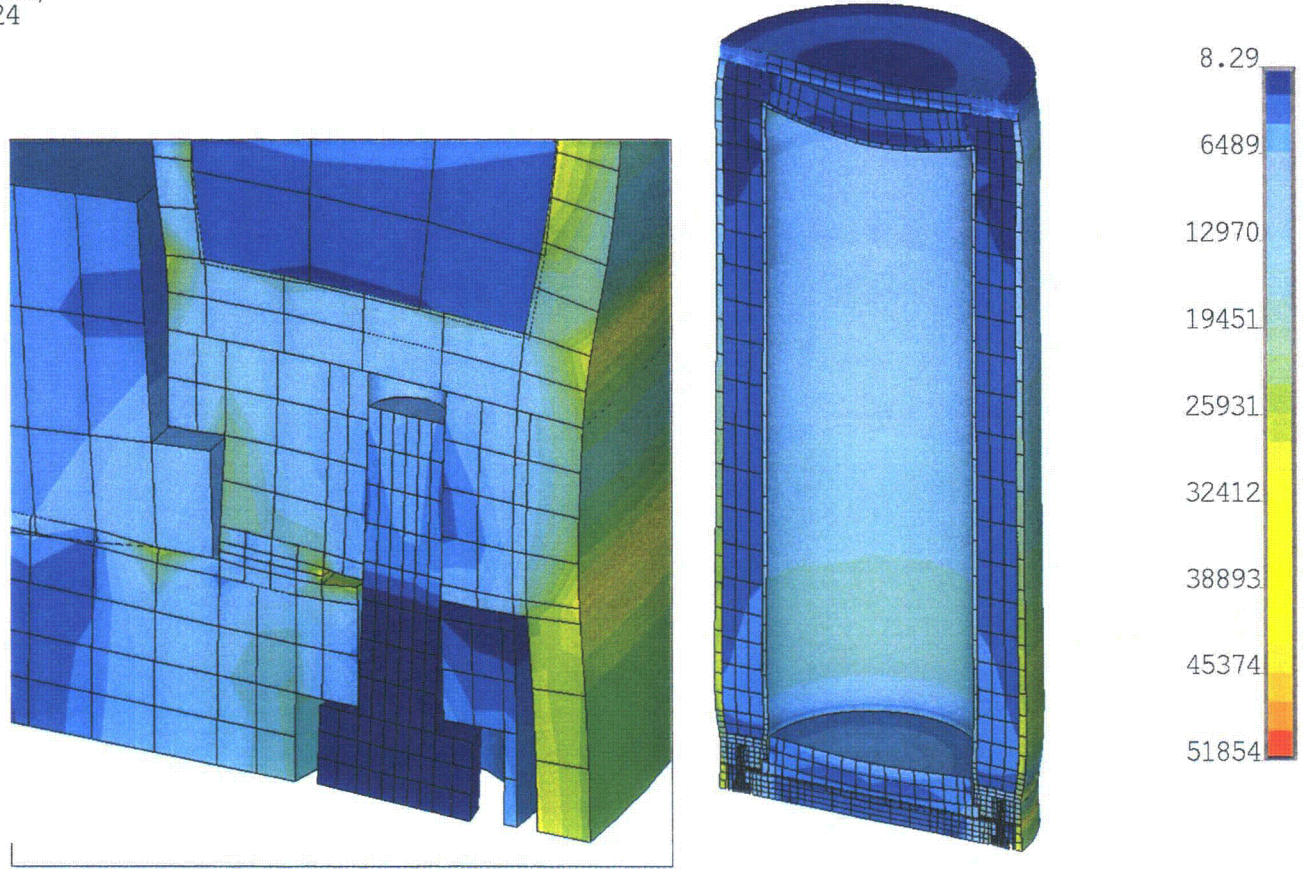
Figure 9  
Temperature Profile and Pressure Distribution Used for 30-ft End Drop – Load Combination No.3



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PLOT NO. 1



30-ft End Drop - Hot Condition

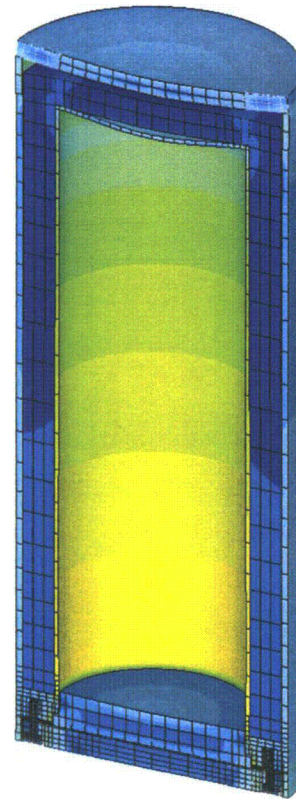
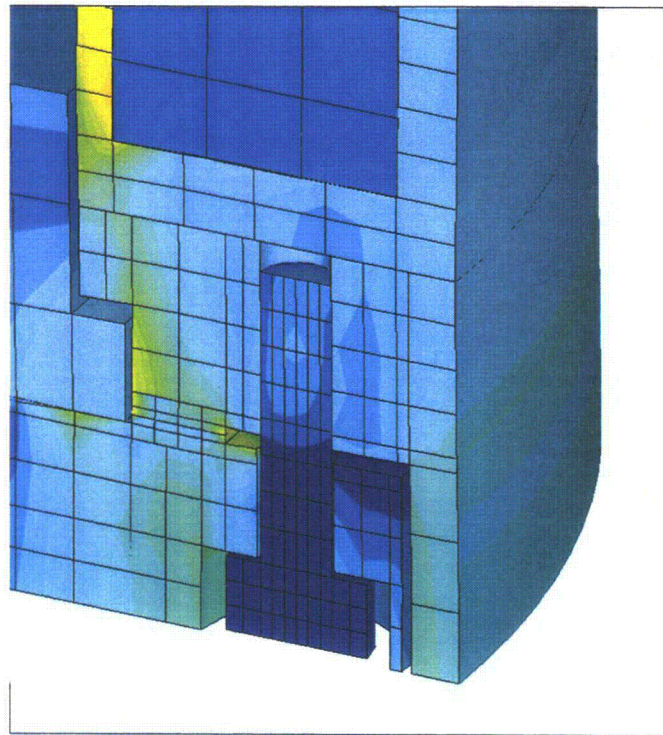
Figure 10

Stress Intensity Plot - 30-ft End Drop - Load Combination No.1

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30-ft End Drop - Cold Condition (Max. Decay Heat)

Figure 11

Stress Intensity Plot - 30-ft End Drop - Load Combination No.2

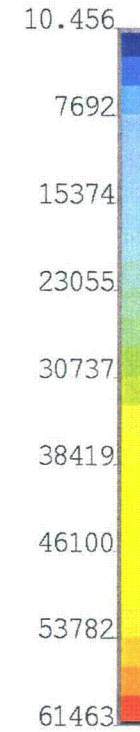
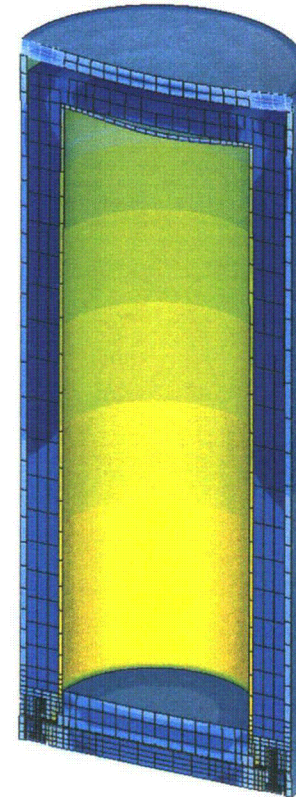
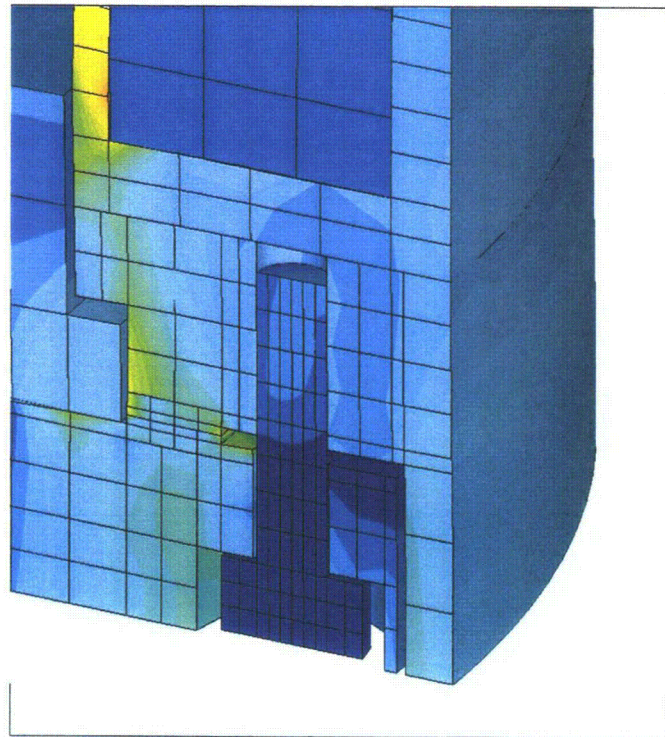


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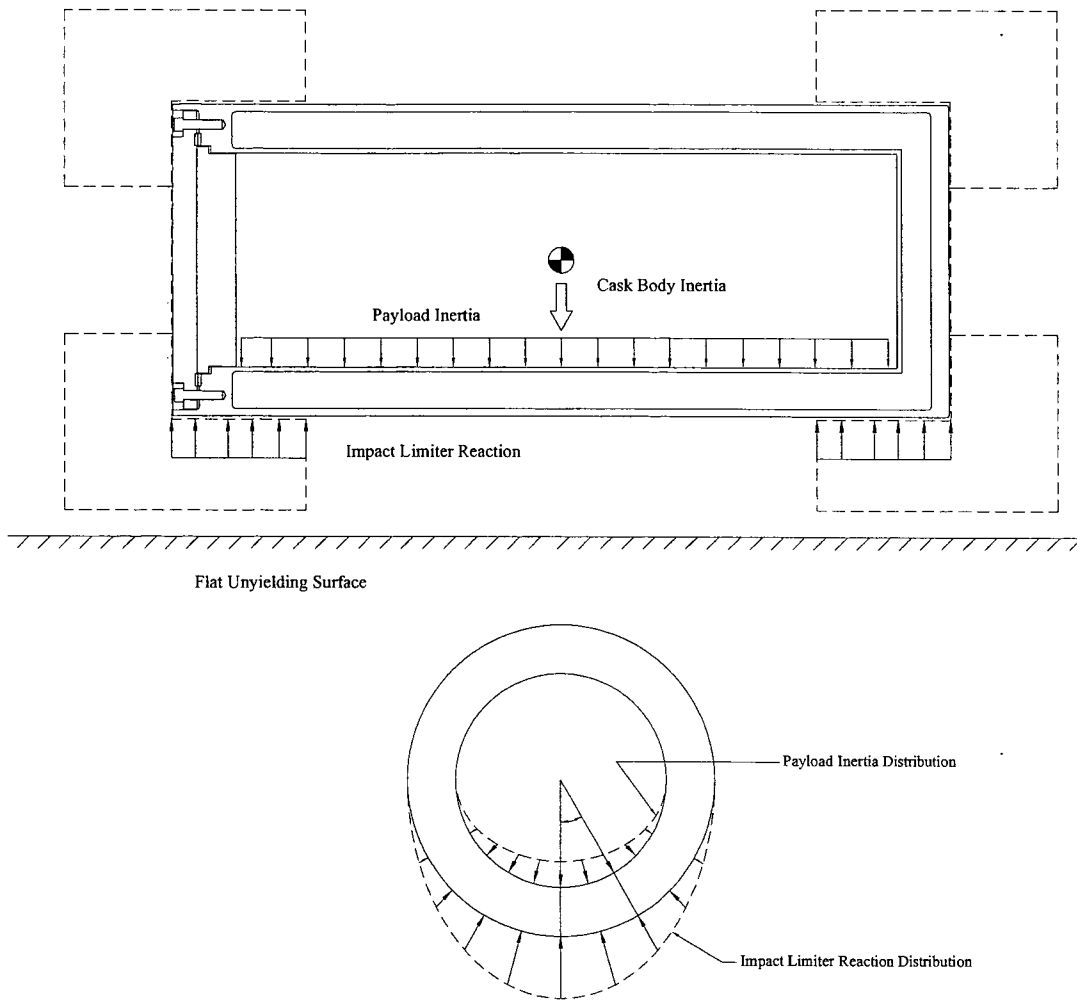
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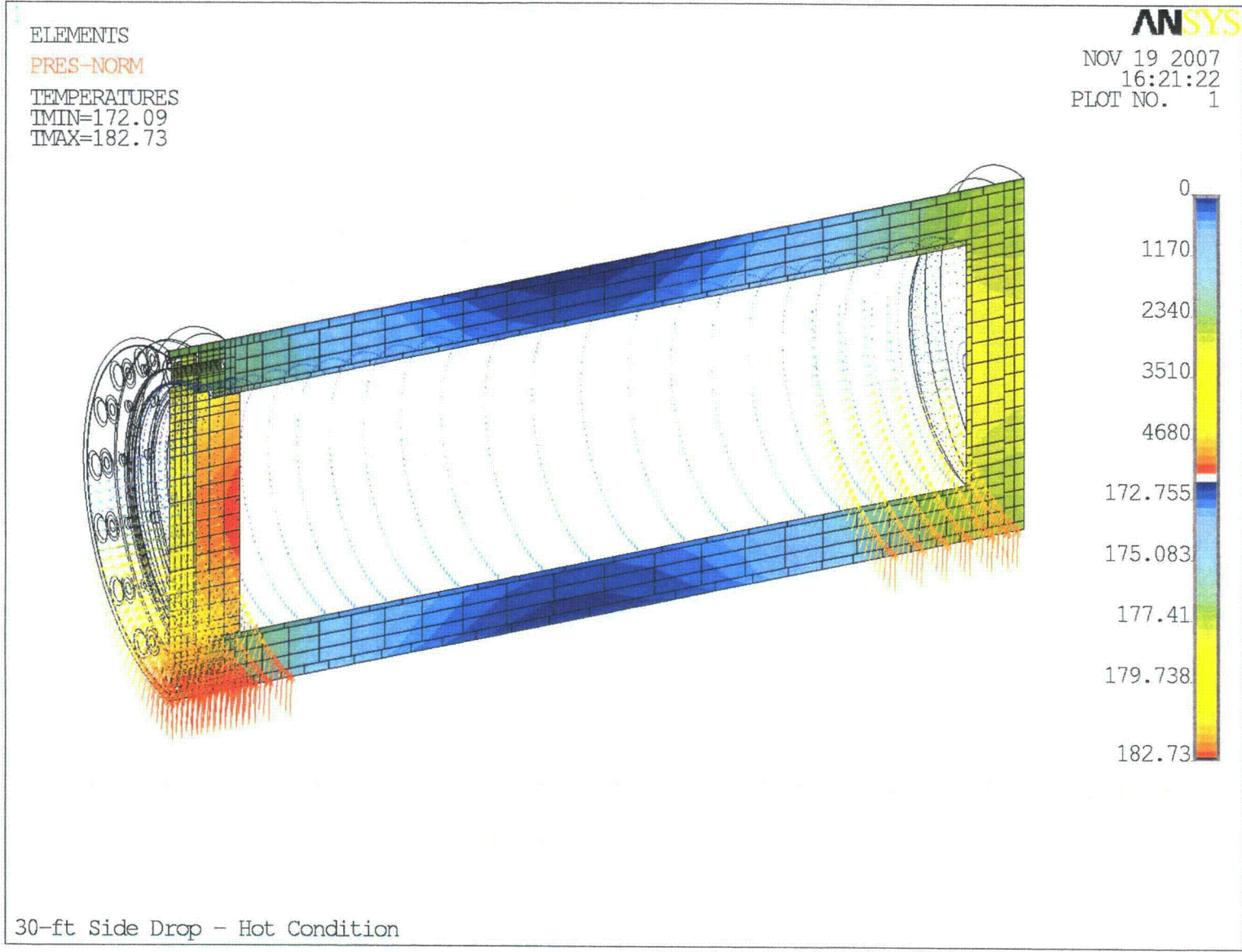
30-ft End Drop - Cold Condition (No Decay Heat)

Figure 12

Stress Intensity Plot - 30-ft End Drop - Load Combination No.3



**Figure 13**  
Load Distribution on the Model during Side Drop



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Figure 14  
 Temperature Profile and Pressure Distribution Used for 30-ft Side Drop – Load Combination No.1



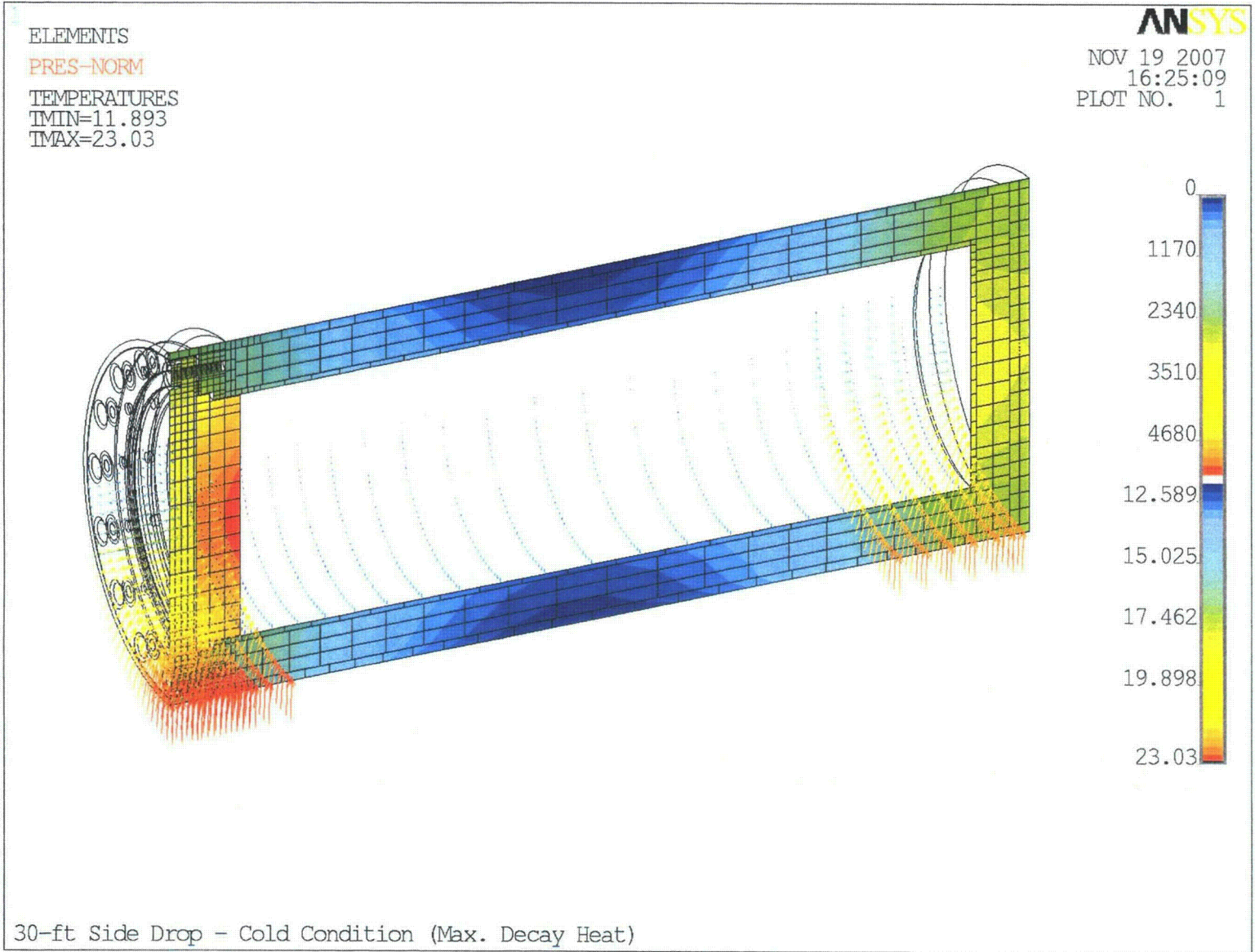
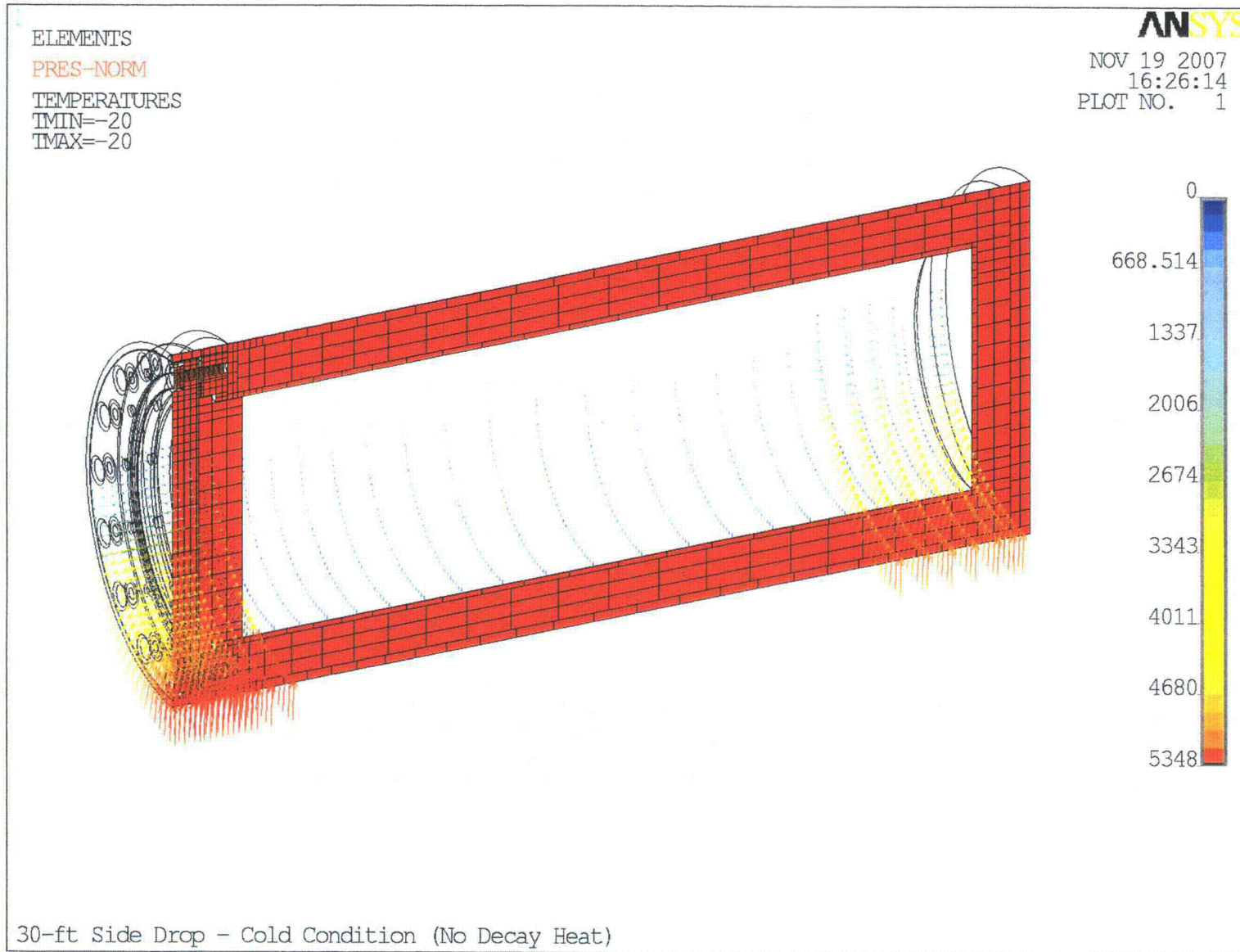


Figure 15  
 Temperature Profile and Pressure Distribution Used for 30-ft Side Drop – Load Combination No.2



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Figure 16  
 Temperature Profile and Pressure Distribution Used for 30-ft Side Drop - Load Combination No.3



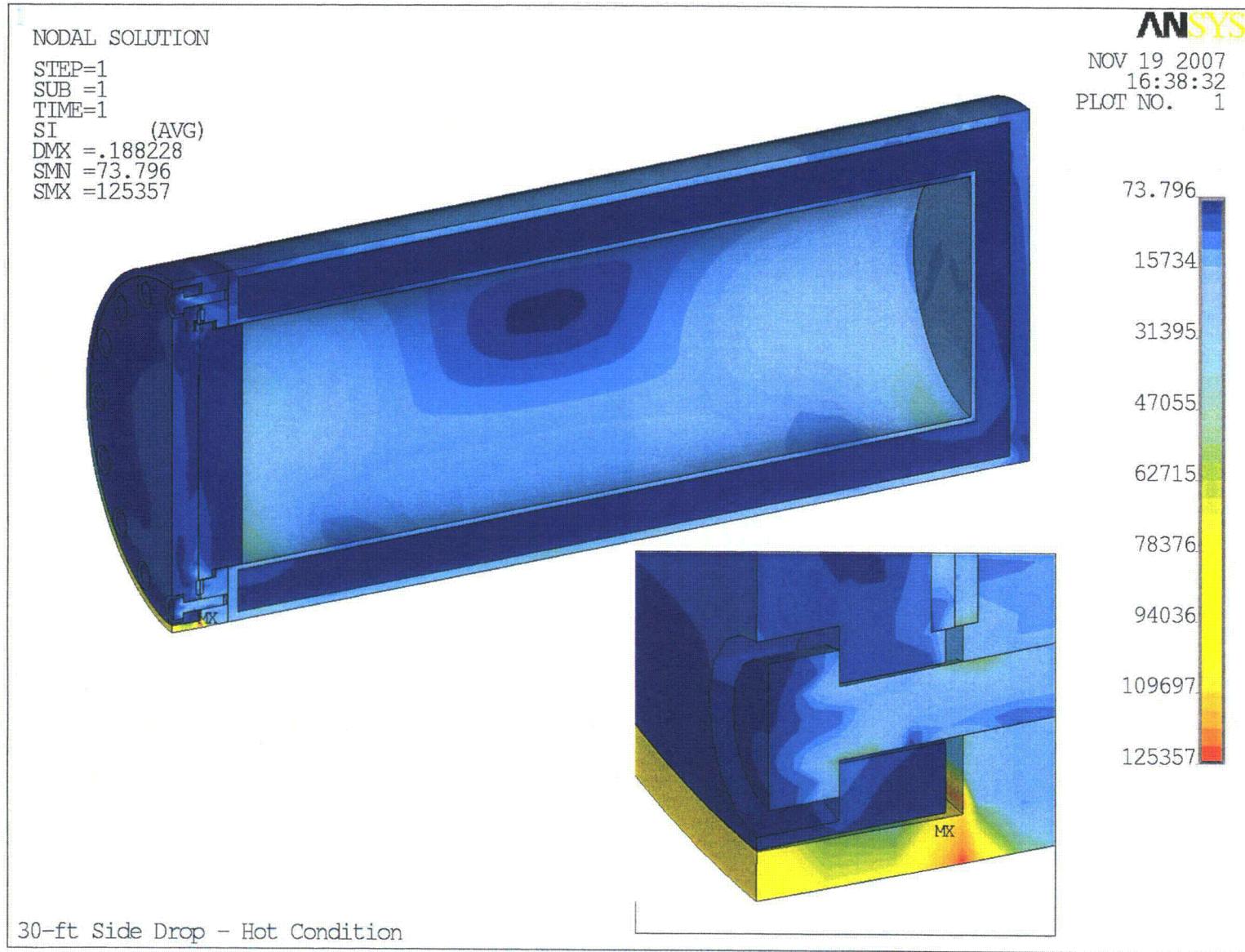


Figure 17  
Stress Intensity Plot – 30-ft Side Drop – Load Combination No.1



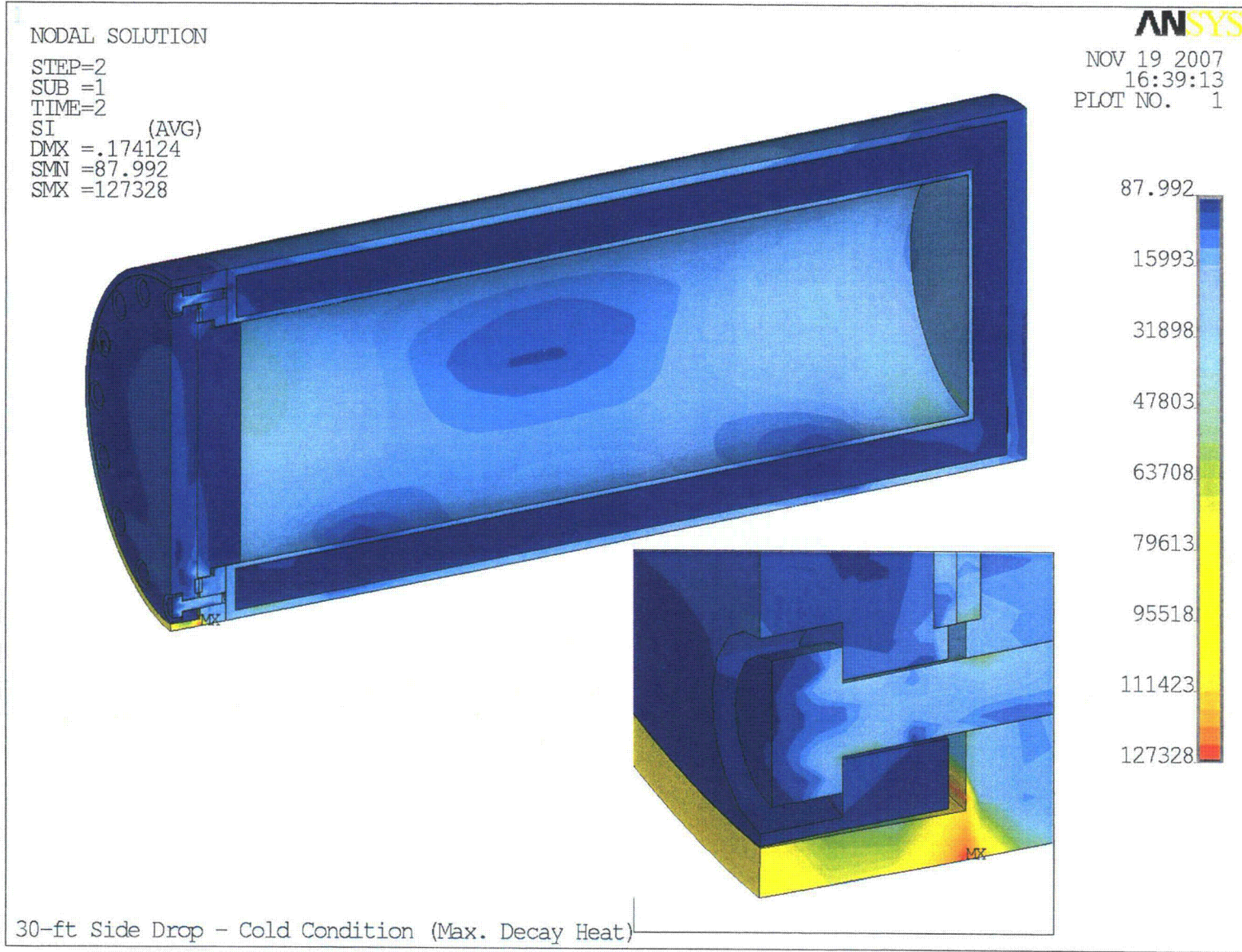


Figure 18  
Stress Intensity Plot - 30-ft Side Drop - Load Combination No.2

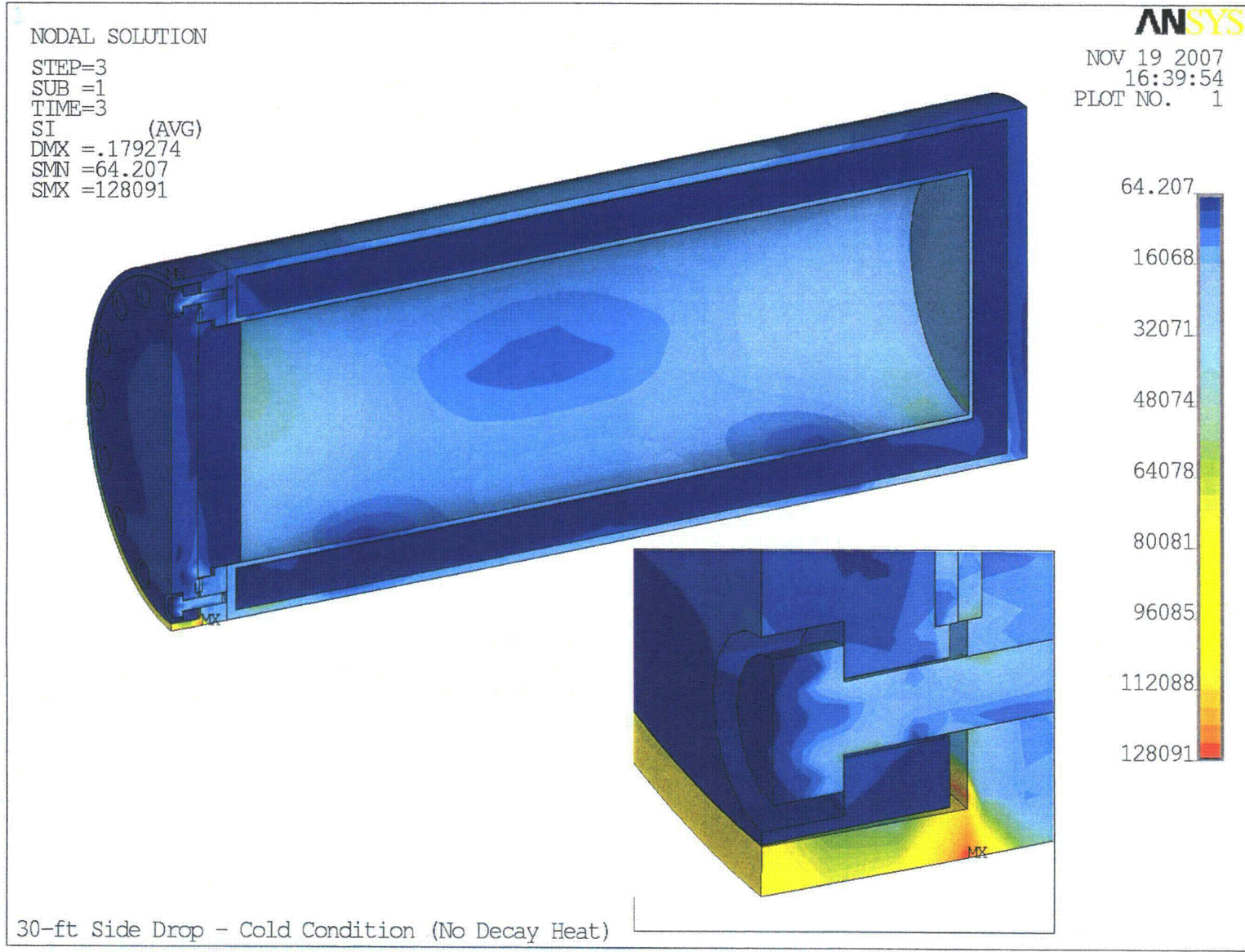


Figure 19  
Stress Intensity Plot – 30-ft Side Drop – Load Combination No.3



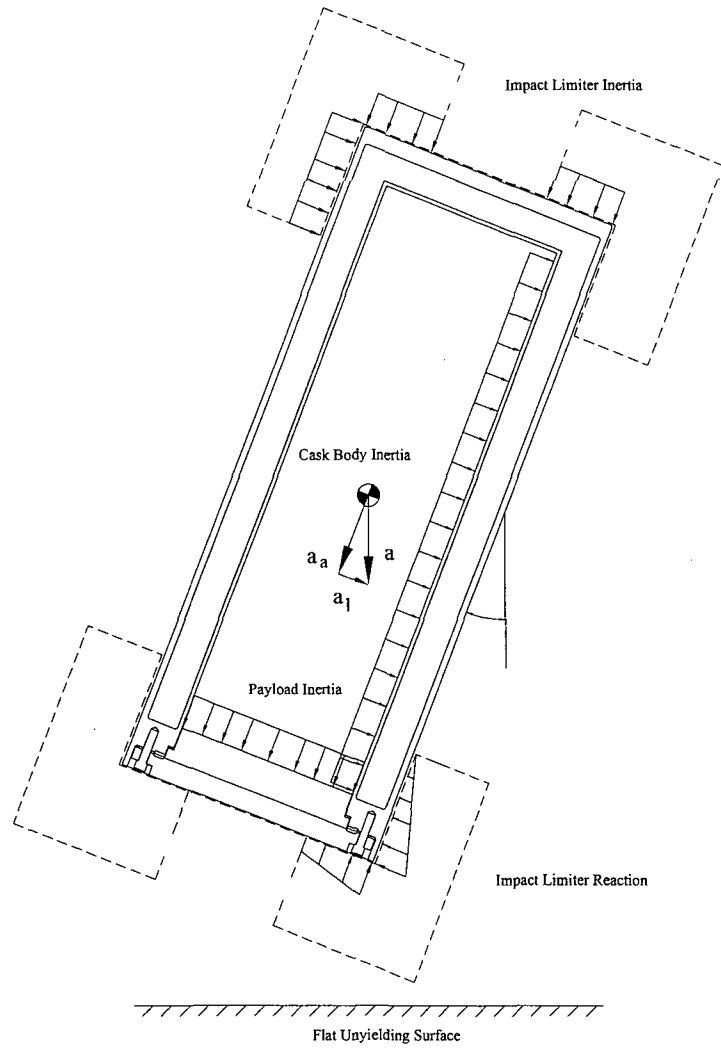


Figure 20  
Load Distribution on the Model during Corner Drop

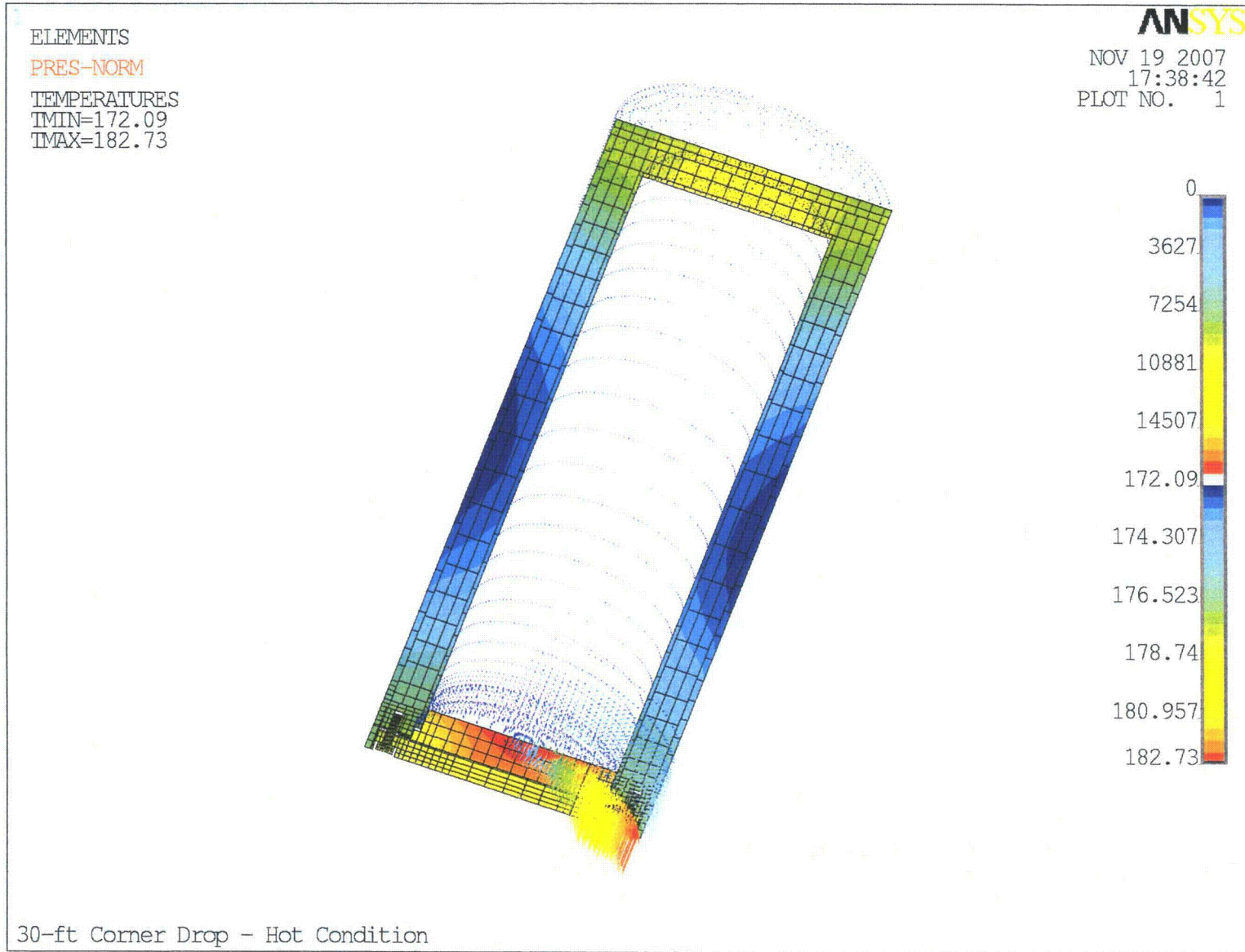


Figure 21  
 Temperature Profile and Pressure Distribution Used for 30-ft Corner Drop – Load Combination No.1

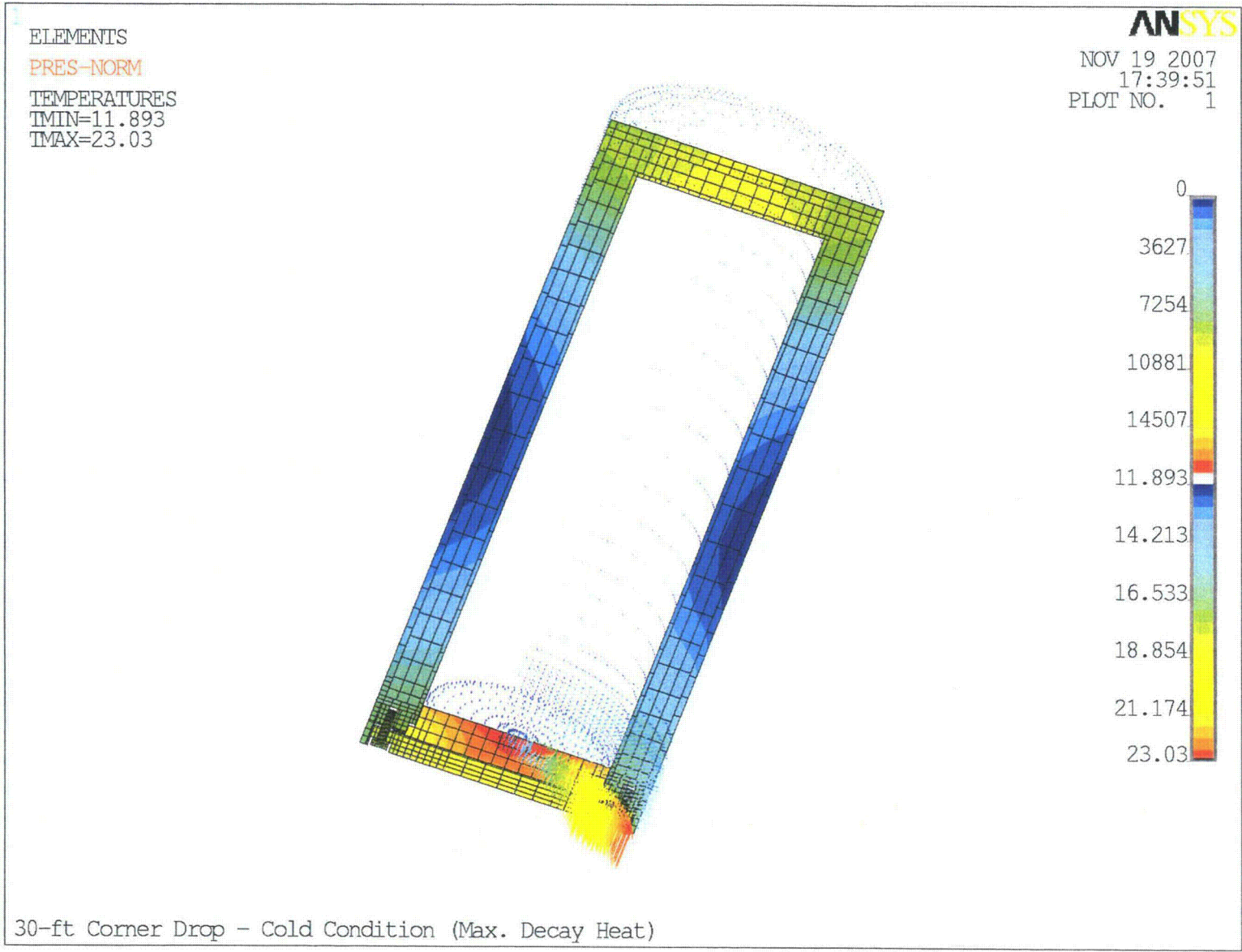


Figure 22

Temperature Profile and Pressure Distribution Used for 30-ft Corner Drop - Load Combination No.2

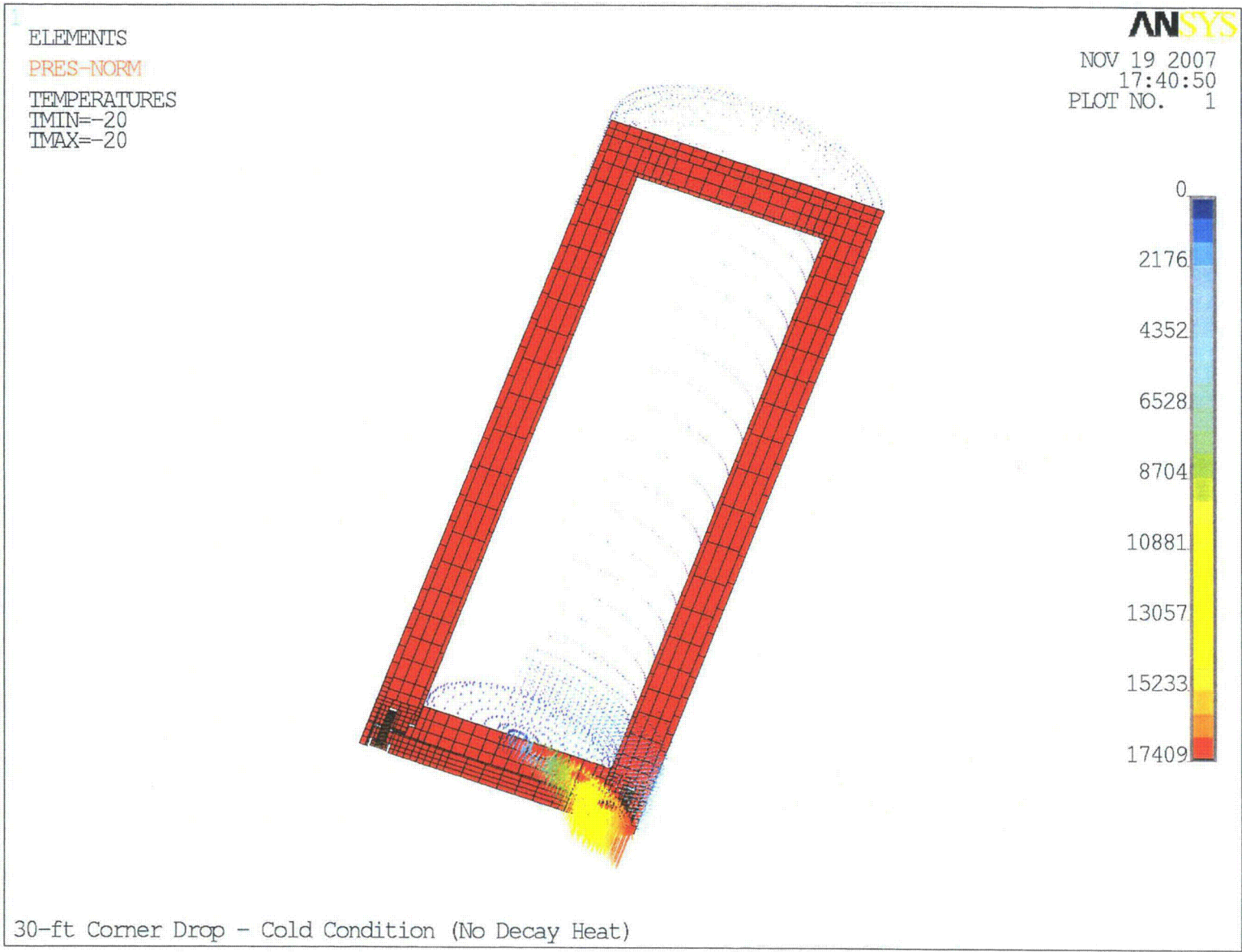


Figure 23

Temperature Profile and Pressure Distribution Used for 30-ft Corner Drop – Load Combination No.3



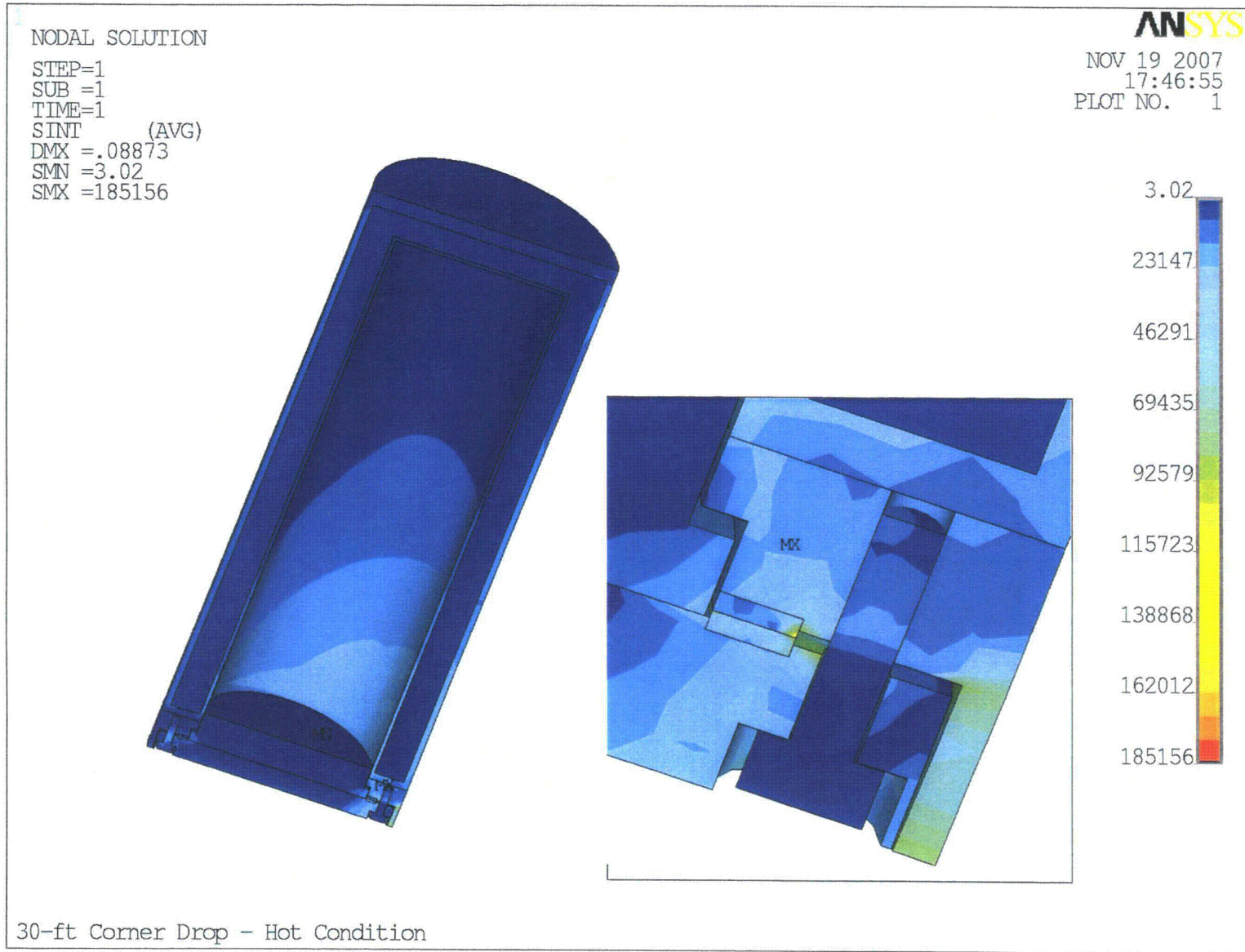


Figure 24  
Stress Intensity Plot – 30-ft Corner Drop – Load Combination No.1

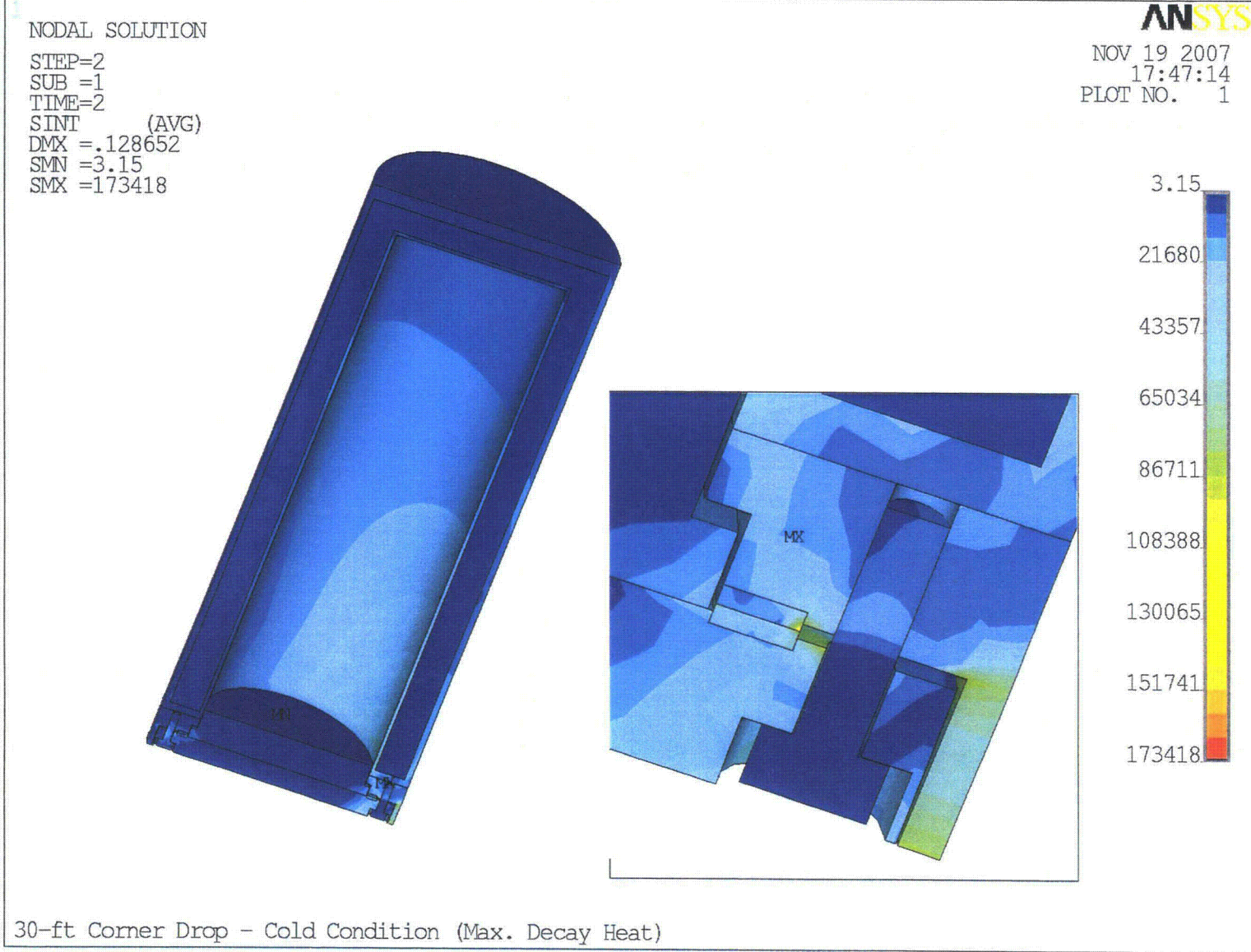


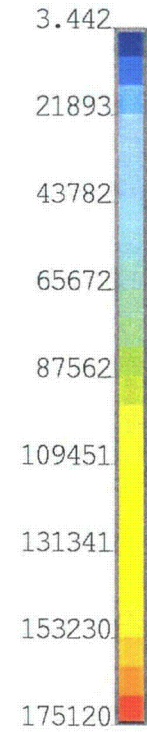
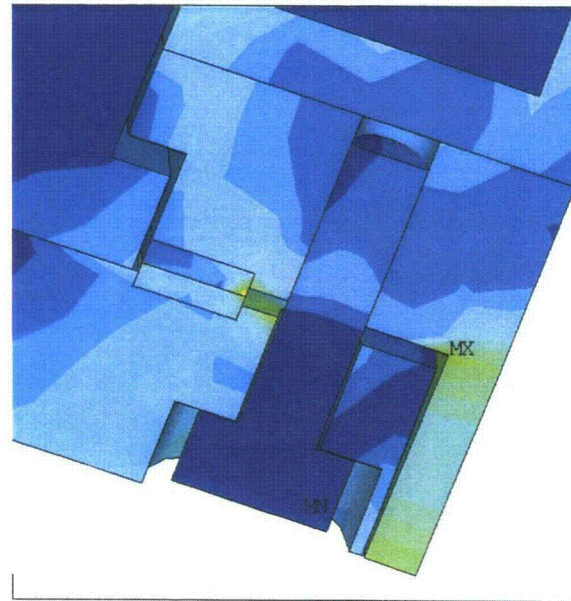
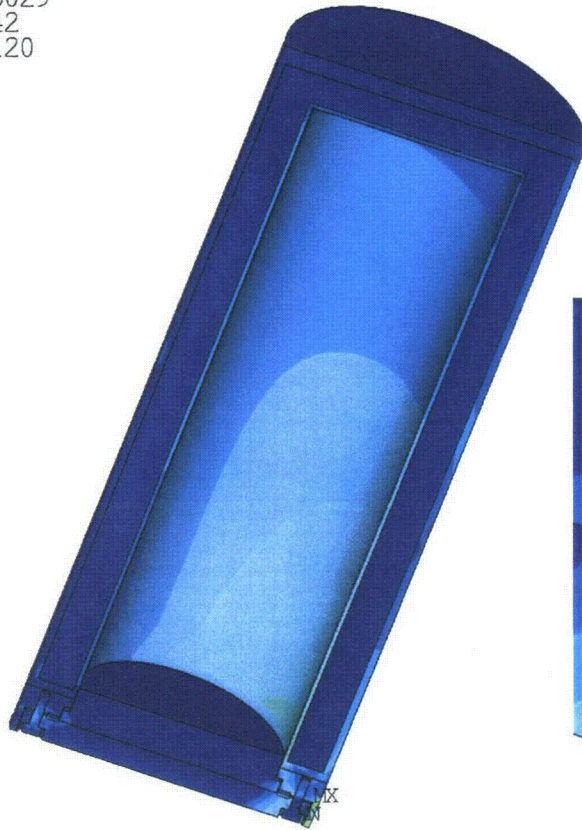
Figure 25  
Stress Intensity Plot - 30-ft Corner Drop - Load Combination No.2



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PLOT NO. 1



30-ft Corner Drop - Cold Condition (No Decay Heat)

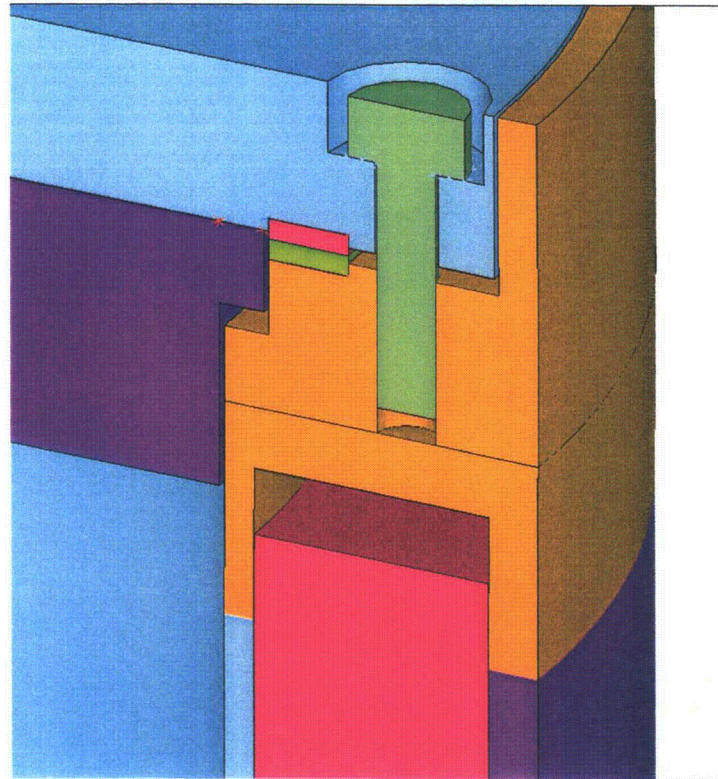
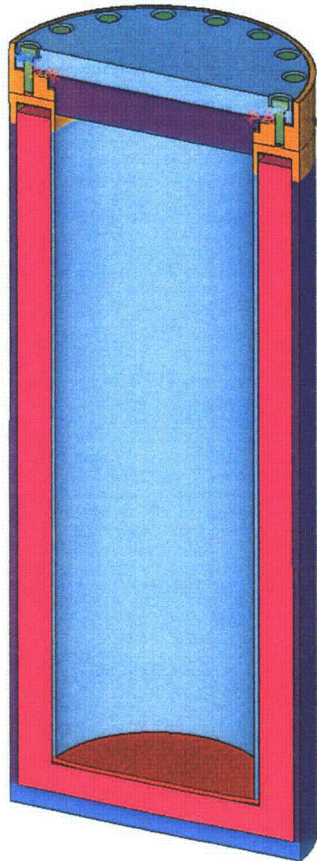
Figure 26  
Stress Intensity Plot - 30-ft Corner Drop - Load Combination No.3

DISPLACEMENT

STEP=1  
SUB =1  
TIME=1  
DMX =.32559

ANSYS

NOV 9 2007  
12:04:50  
PLOT NO. 1



Maximum Relative  
Displacement at the Bolting  
Ring-Lead Interface = 0.3172 in

30-ft Bottom End Drop - Cold Condition (No Decay Heat)

Title \_\_\_\_\_ Structural Analyses of the 3-60B Cask Under Drop Conditions  
Calc. No. ST-504 (Figures) \_\_\_\_\_ Rev. 1 \_\_\_\_\_ Sheet 27 of 45

Figure 27

Displacements During 30-ft End Drop on the Bottom



ELEMENTS

PRES-NORM

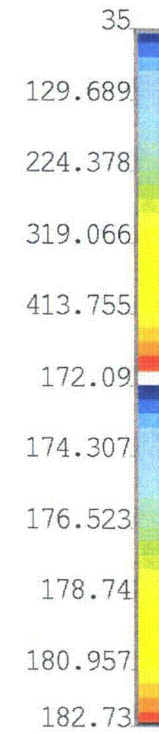
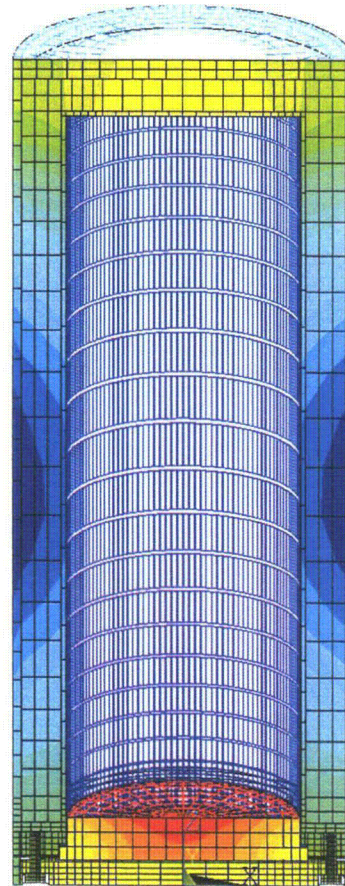
TEMPERATURES

TMIN=172.09

TMAX=182.73

ANSYS

NOV 20 2007  
09:07:01  
PLOT NO. 1

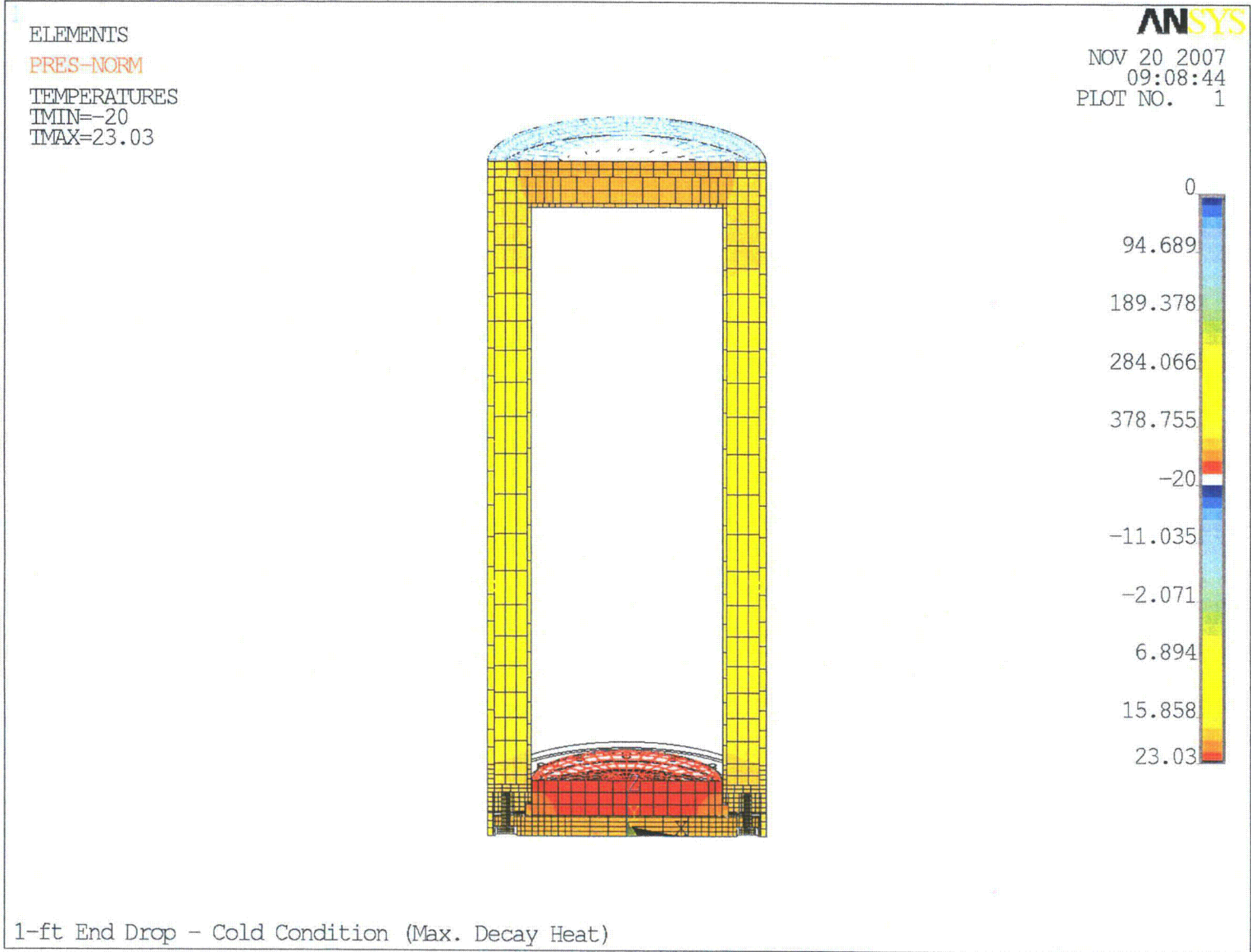


1-ft End Drop - Hot Condition

Title \_\_\_\_\_ Structural Analyses of the 3-60B Cask Under Drop Conditions  
Calc. No. ST-504 (Figures) \_\_\_\_\_ Rev. 1 \_\_\_\_\_ Sheet 28 of 45

Figure 28

Temperature Profile and Pressure Distribution Used for 1-ft End Drop - Load Combination No.1



Title \_\_\_\_\_ Structural Analyses of the 3-60B Cask Under Drop Conditions  
 Calc. No. ST-504 (Figures) \_\_\_\_\_ Rev. 1 \_\_\_\_\_ Sheet 29 of 45

Figure 29  
 Temperature Profile and Pressure Distribution Used for 1-ft End Drop – Load Combination No.2

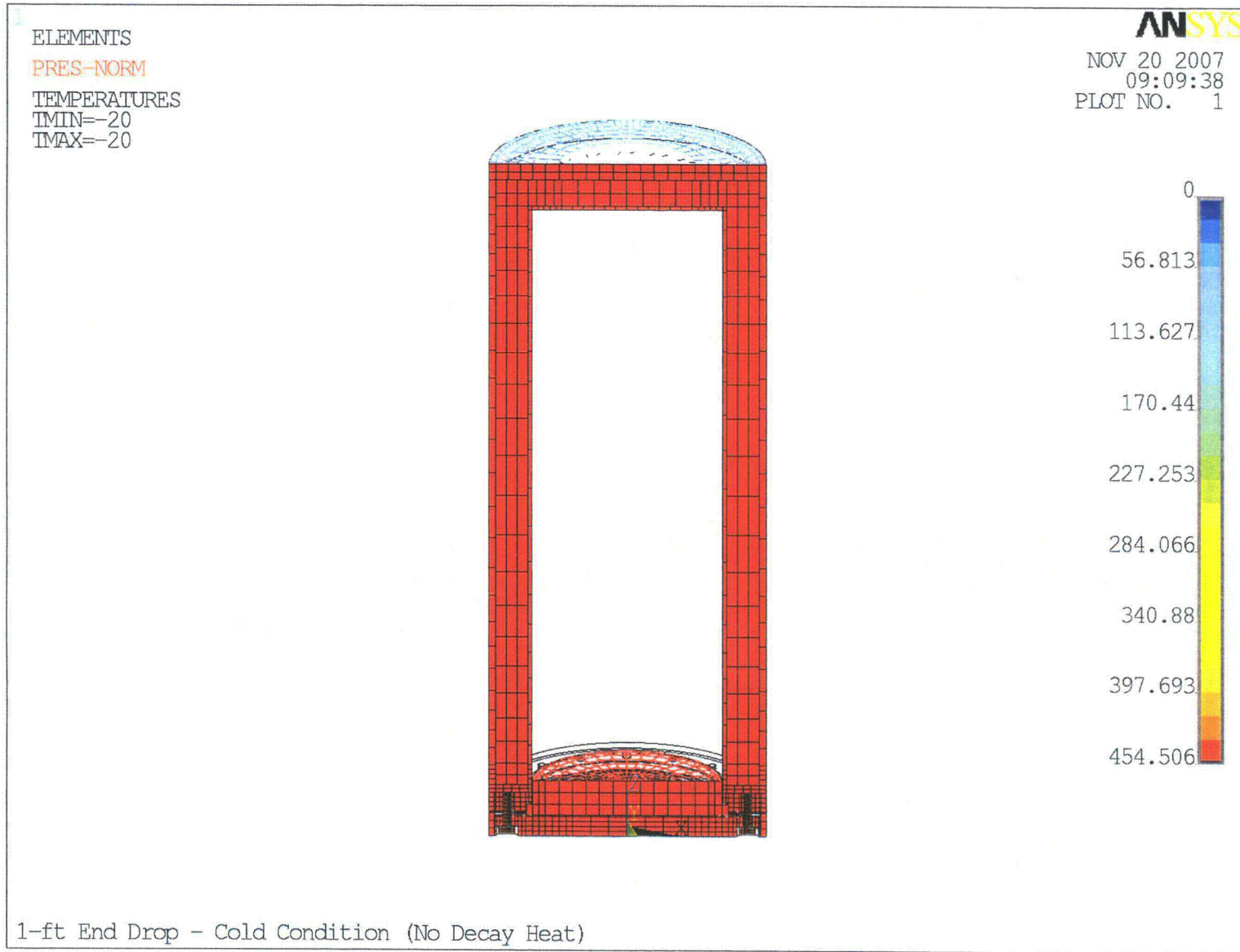


Figure 30

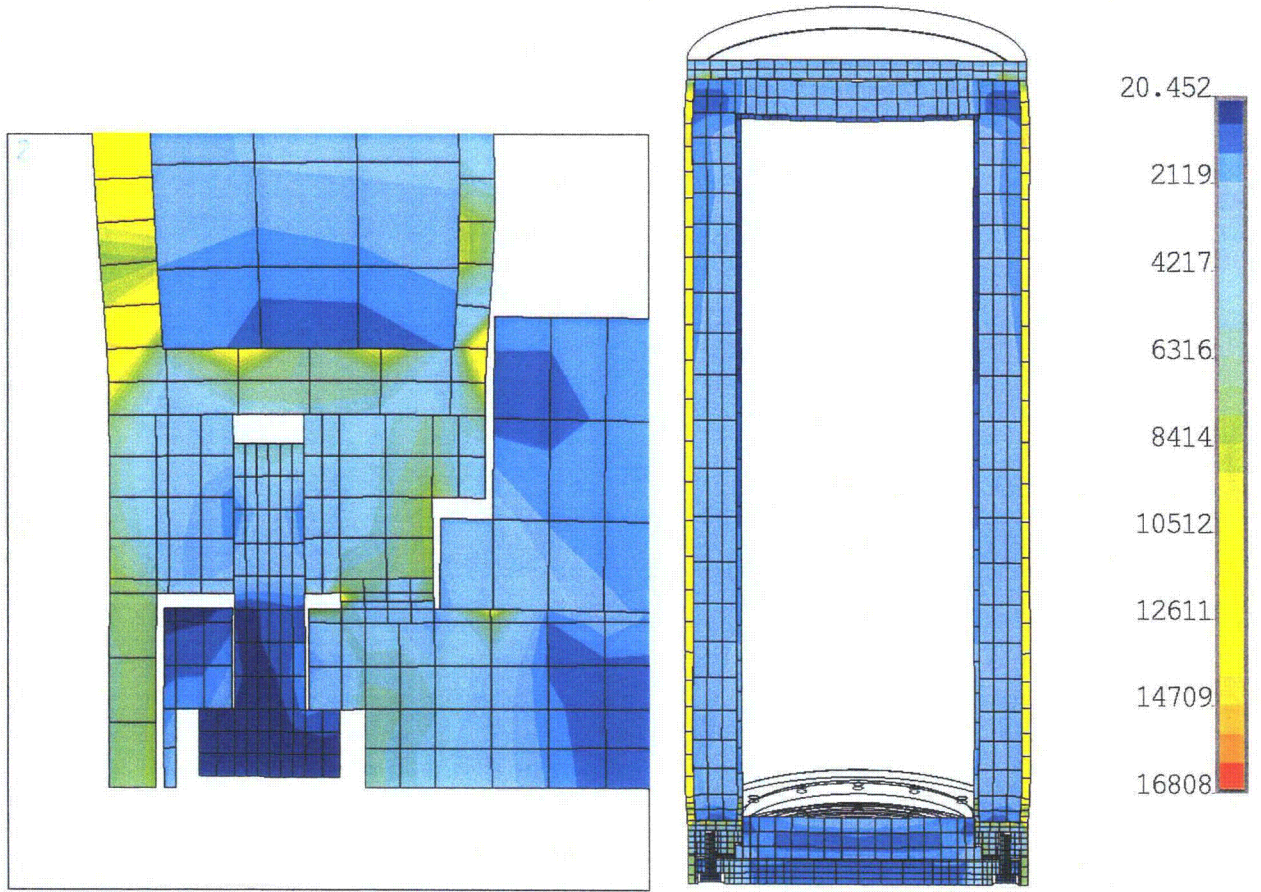
Temperature Profile and Pressure Distribution Used for 1-ft End Drop – Load Combination No.3



NODAL SOLUTION

STEP=1  
SUB =3  
TIME=1  
SI (AVG)  
DMX =.122248  
SMN =20.452  
SMX =16808

ANSYS  
NOV 20 2007  
09:23:57  
PLOT NO. 1



1-ft End Drop - Hot Condition

Figure 31

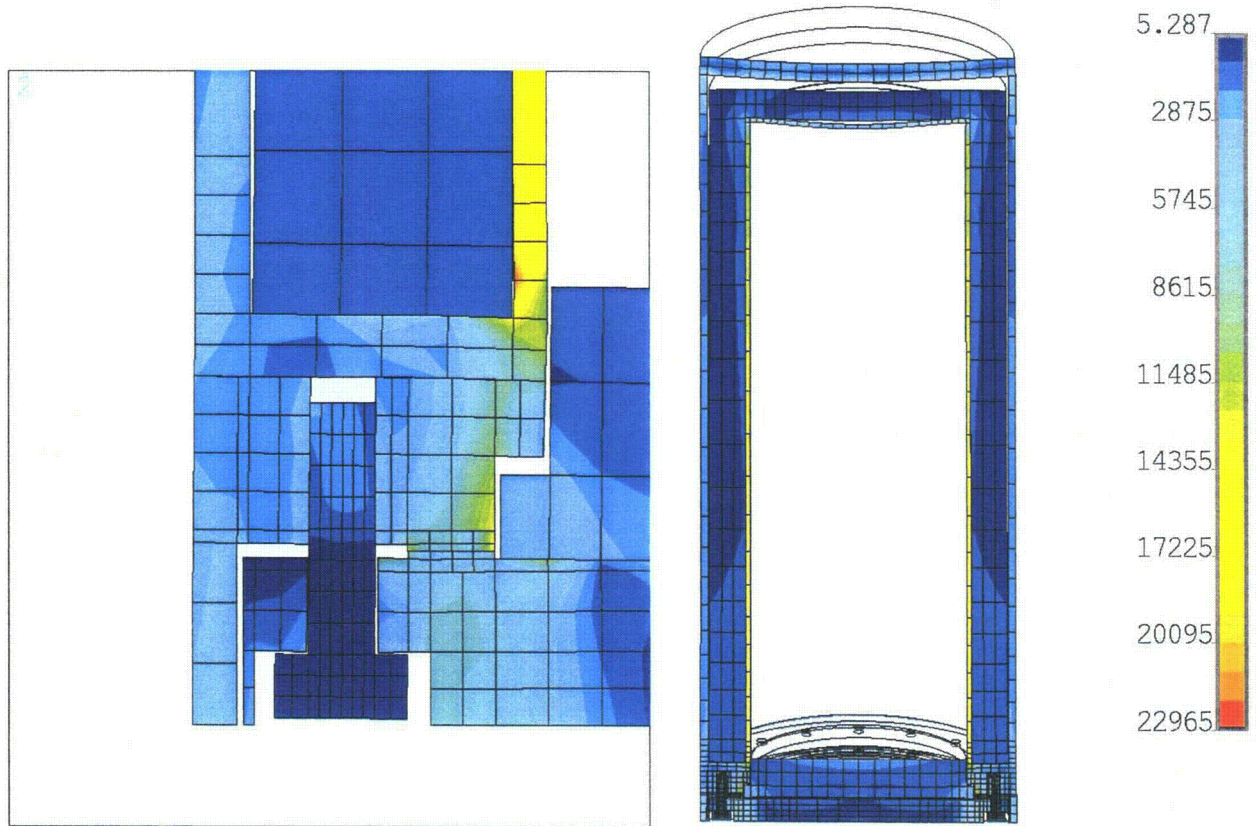
Stress Intensity Plot – 1-ft End Drop – Load Combination No.1

NODAL SOLUTION

STEP=2  
SUB =3  
TIME=2  
SI (AVG)  
DMX =.13075  
SMN =5.287  
SMX =22965

ANSYS

NOV 20 2007  
09:26:09  
PLOT NO. 1



1-ft End Drop - Cold Condition (Max. Decay Heat)

Figure 32

Stress Intensity Plot - 1-ft End Drop - Load Combination No.2





Figure 33  
Stress Intensity Plot – 1-ft End Drop – Load Combination No.3



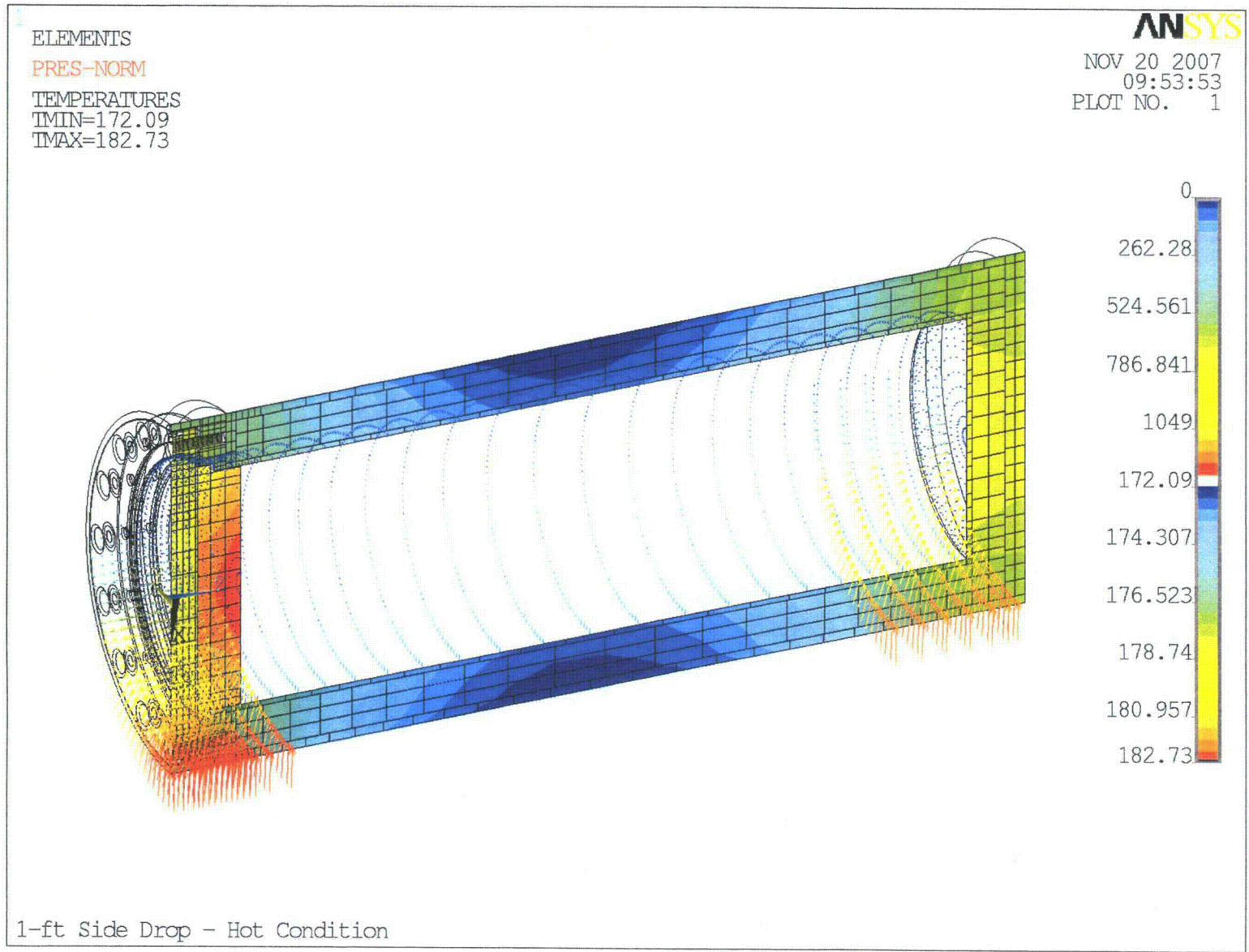


Figure 34

Temperature Profile and Pressure Distribution Used for 1-ft Side Drop – Load Combination No.1

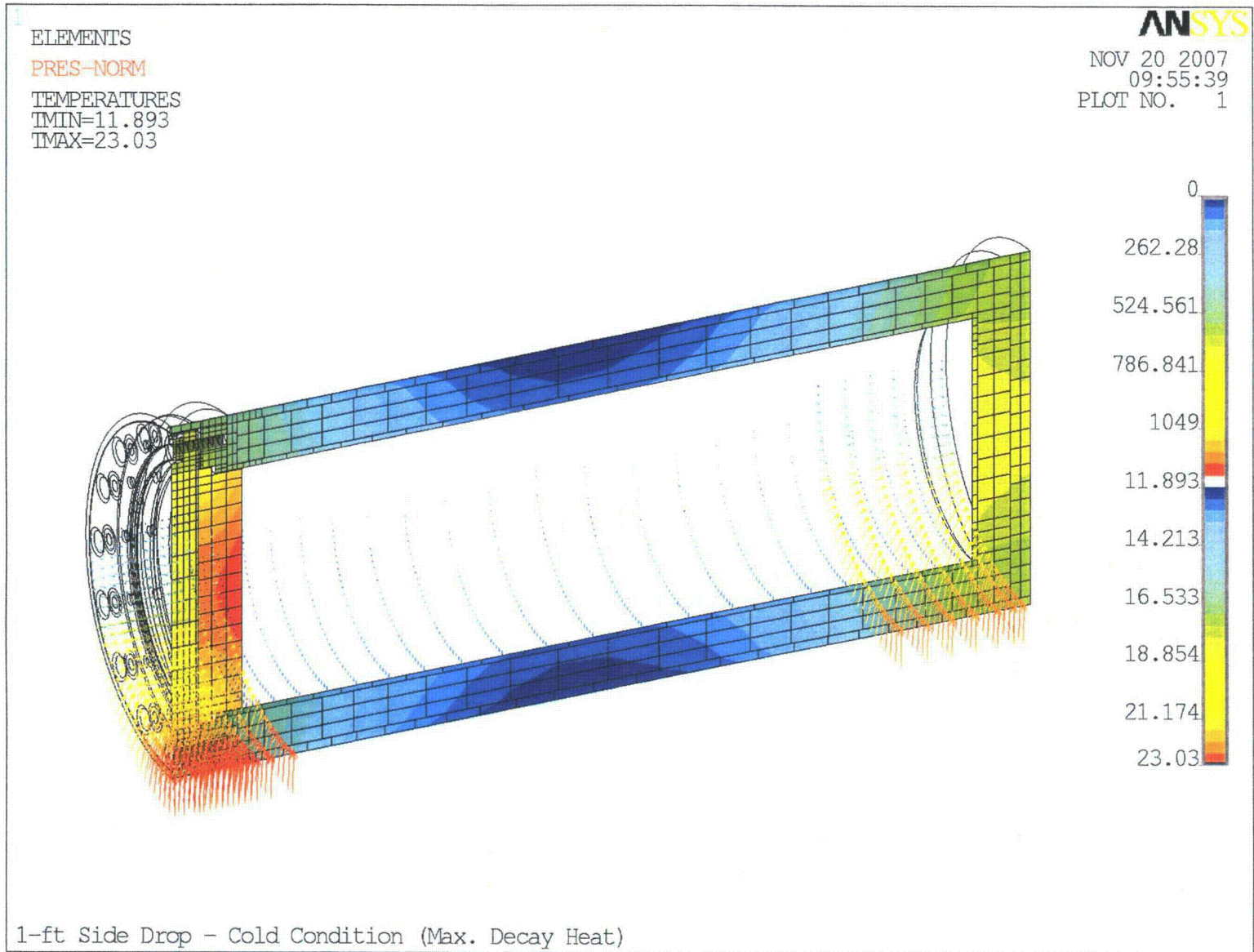
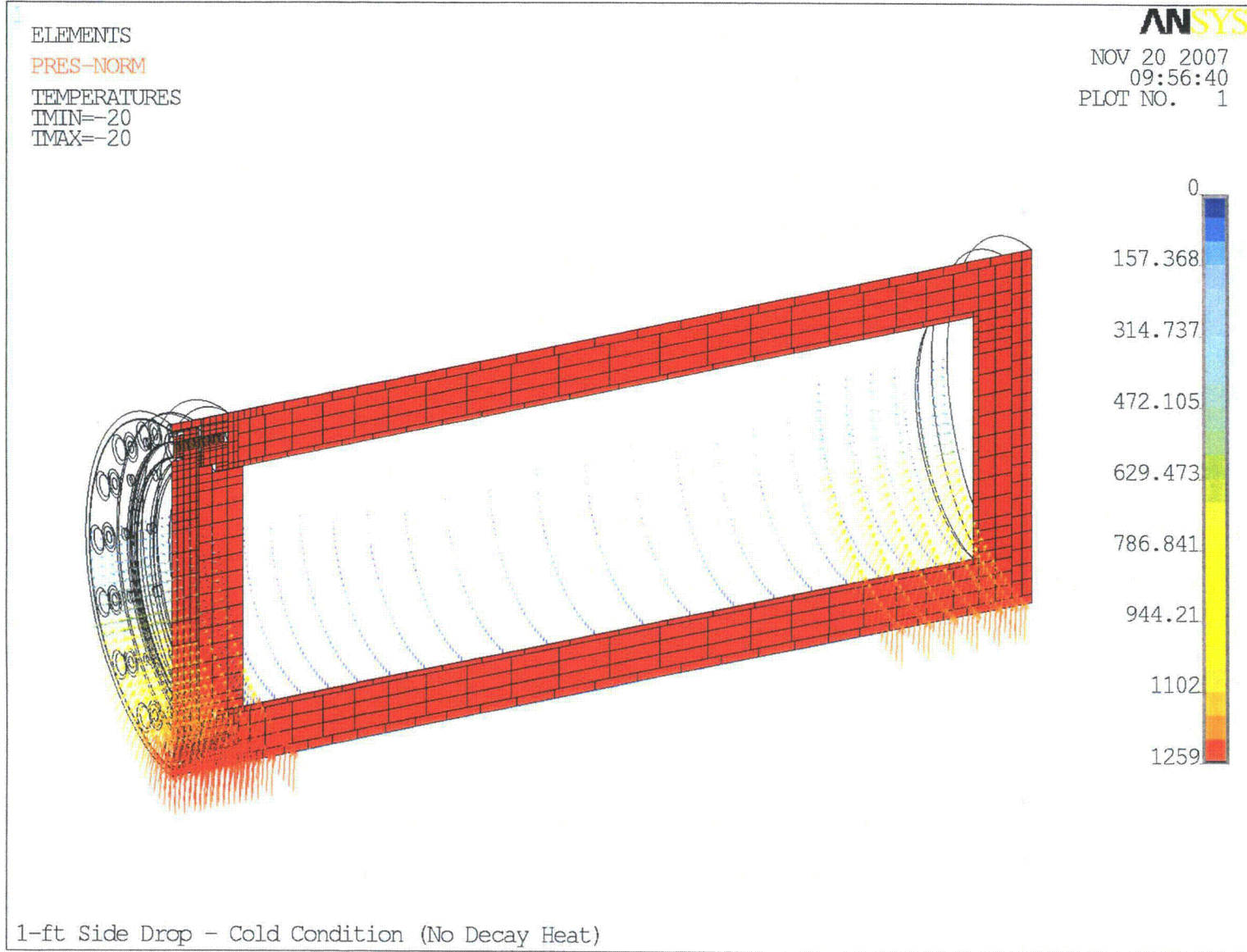


Figure 35

Temperature Profile and Pressure Distribution Used for 1-ft Side Drop – Load Combination No.2





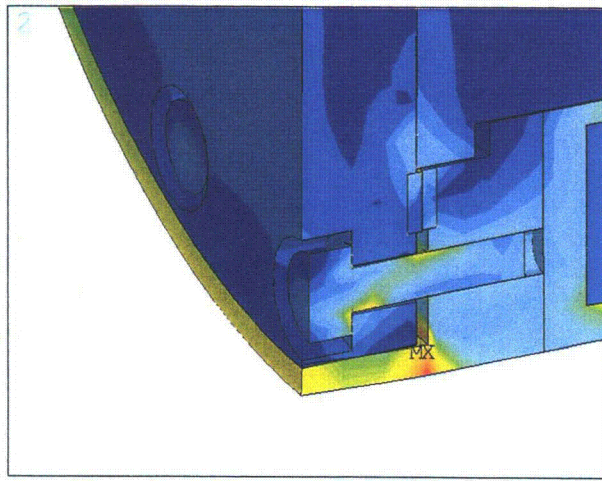
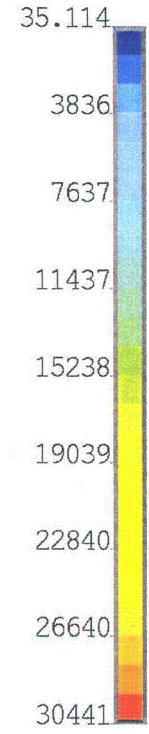
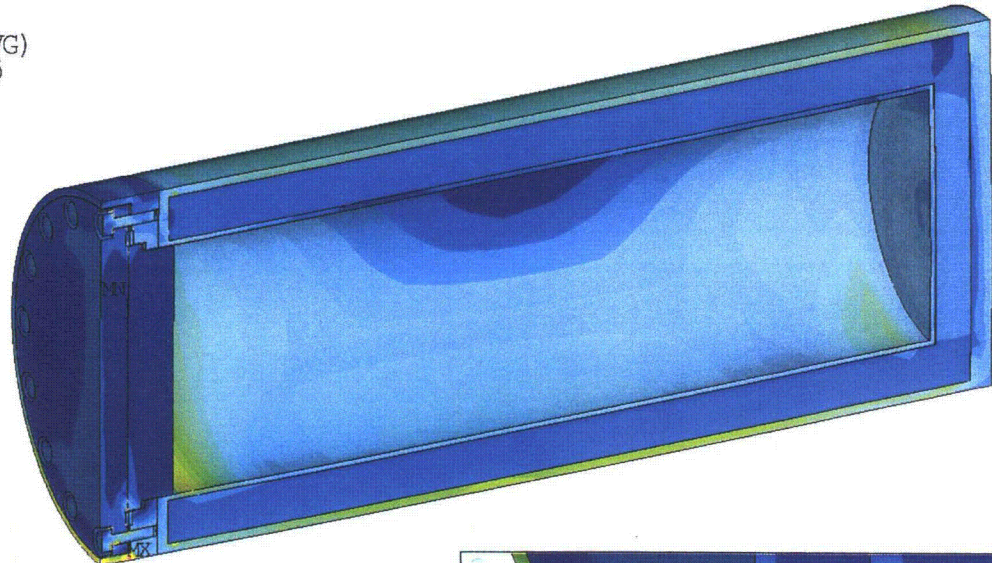
Title \_\_\_\_\_ Structural Analyses of the 3-60B Cask Under Drop Conditions  
 Calc. No. ST-504 (Figures) \_\_\_\_\_ Rev. 1 \_\_\_\_\_ Sheet 36 of 45

Figure 36  
 Temperature Profile and Pressure Distribution Used for 1-ft Side Drop – Load Combination No.3

NODAL SOLUTION

STEP=1  
SUB =1  
TIME=1  
SI (AVG)  
DMX =.092546  
SMN =35.114  
SMX =30441

ANSYS  
NOV 20 2007  
10:09:48  
PLOT NO. 1



1-ft Side Drop - Hot Condition

Title \_\_\_\_\_ Structural Analyses of the 3-60B Cask Under Drop Conditions  
Calc. No. ST-504 (Figures) \_\_\_\_\_ Rev. 1 \_\_\_\_\_ Sheet 37 of 45

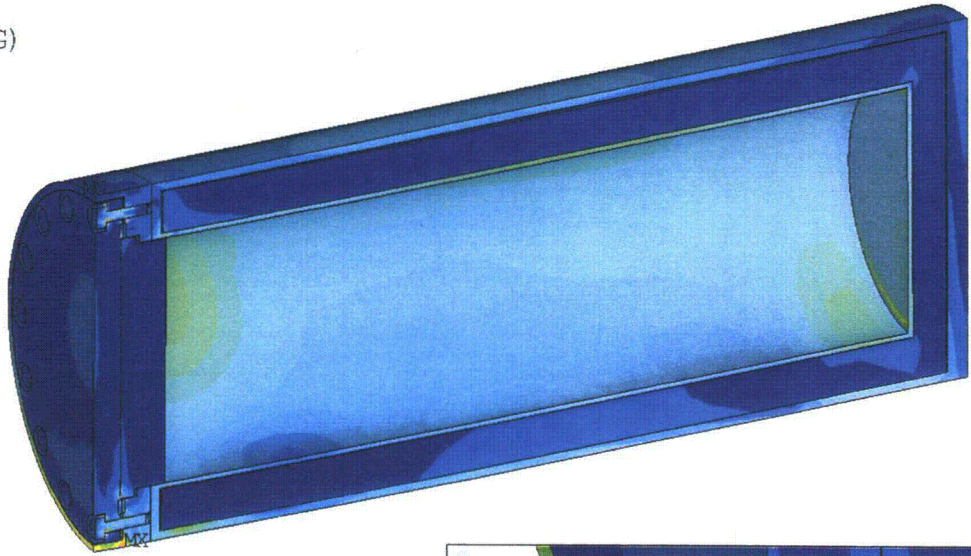
Figure 37  
Stress Intensity Plot - 1-ft Side Drop - Load Combination No.1



NODAL SOLUTION

STEP=2  
SUB =1  
TIME=2  
SI (AVG)  
DMX =.060445  
SMN =17.313  
SMX =31247

**ANSYS**  
NOV 20 2007  
10:11:00  
PLOT NO. 1



1-ft Side Drop - Cold Condition (Max. Decay Heat)

Figure 38

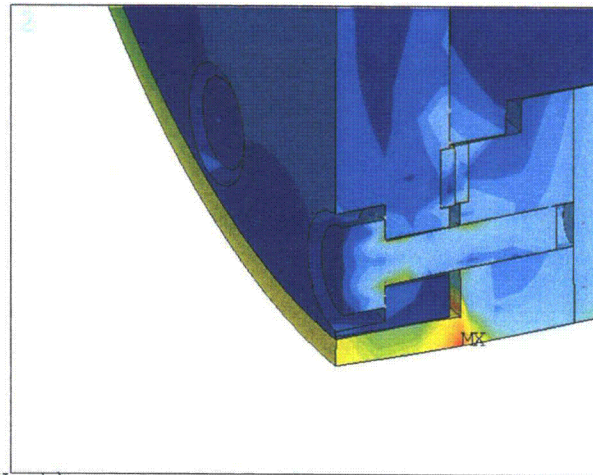
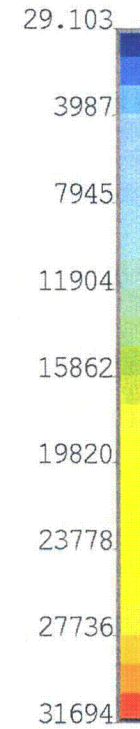
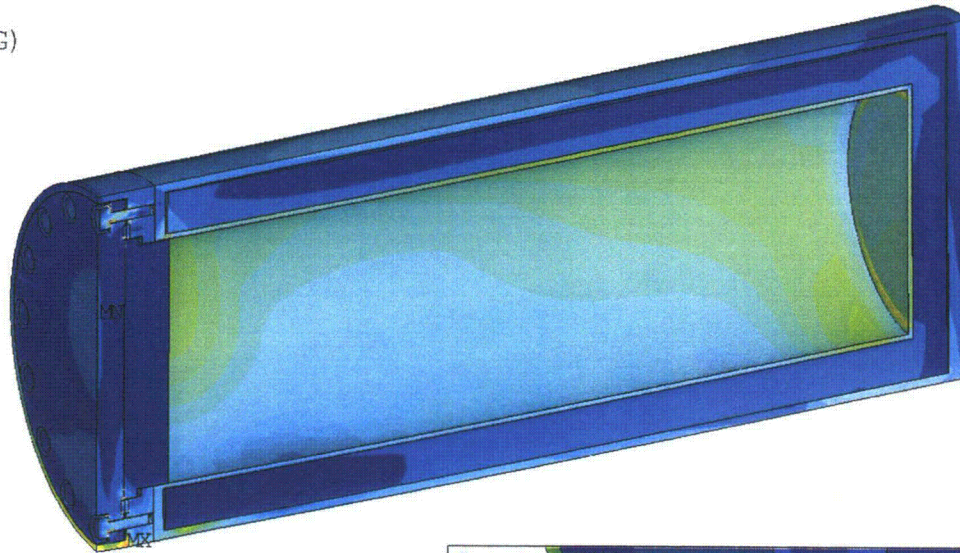
Stress Intensity Plot - 1-ft Side Drop - Load Combination No.2

NODAL SOLUTION

STEP=3  
SUB =1  
TIME=3  
SI (AVG)  
DMX =.093001  
SMN =29.103  
SMX =31694

ANSYS

NOV 20 2007  
10:11:36  
PLOT NO. 1



1-ft Side Drop - Cold Condition (No Decay Heat)

Figure 39

Stress Intensity Plot - 1-ft Side Drop - Load Combination No.3



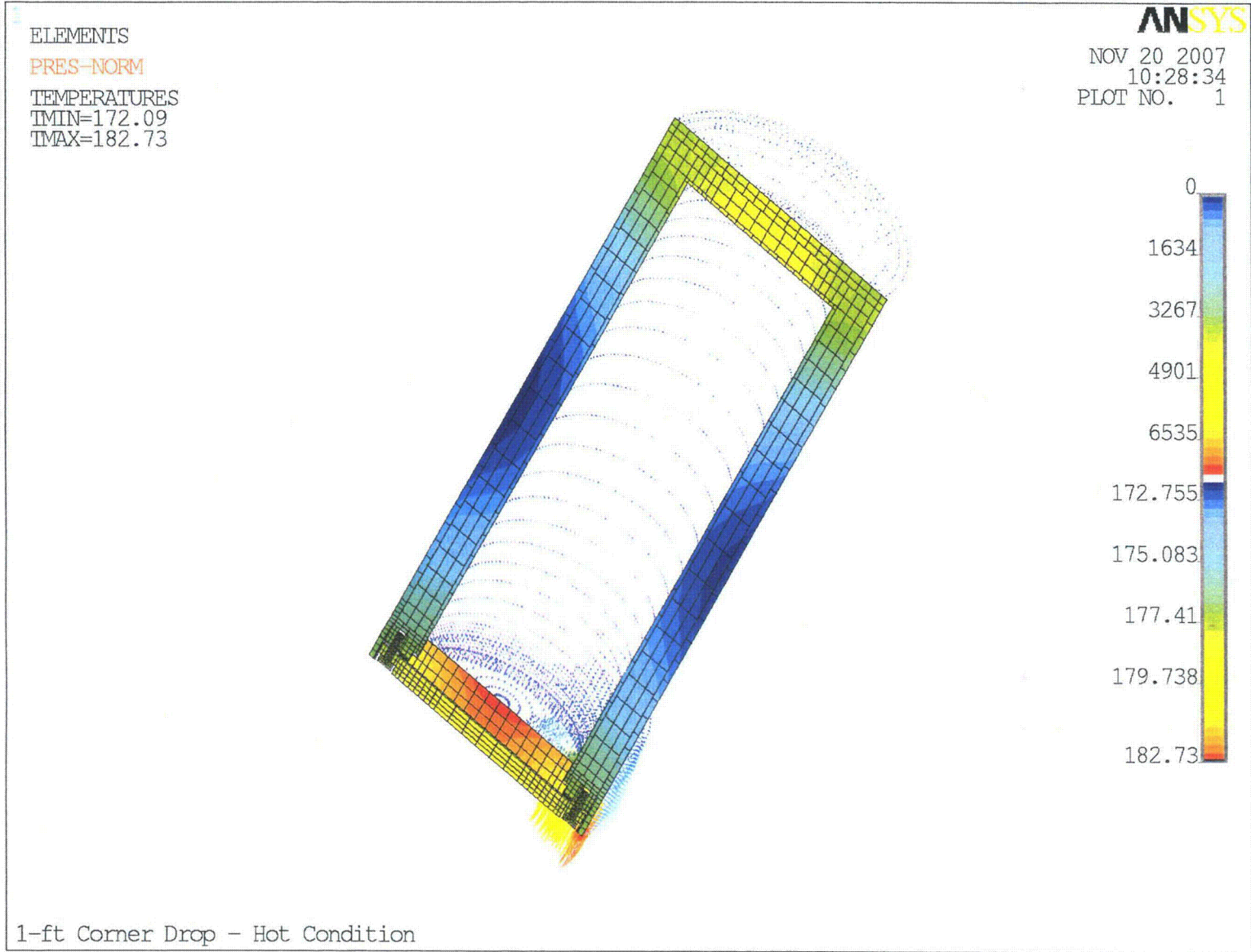


Figure 40

Temperature Profile and Pressure Distribution Used for 1-ft Corner Drop – Load Combination No.1

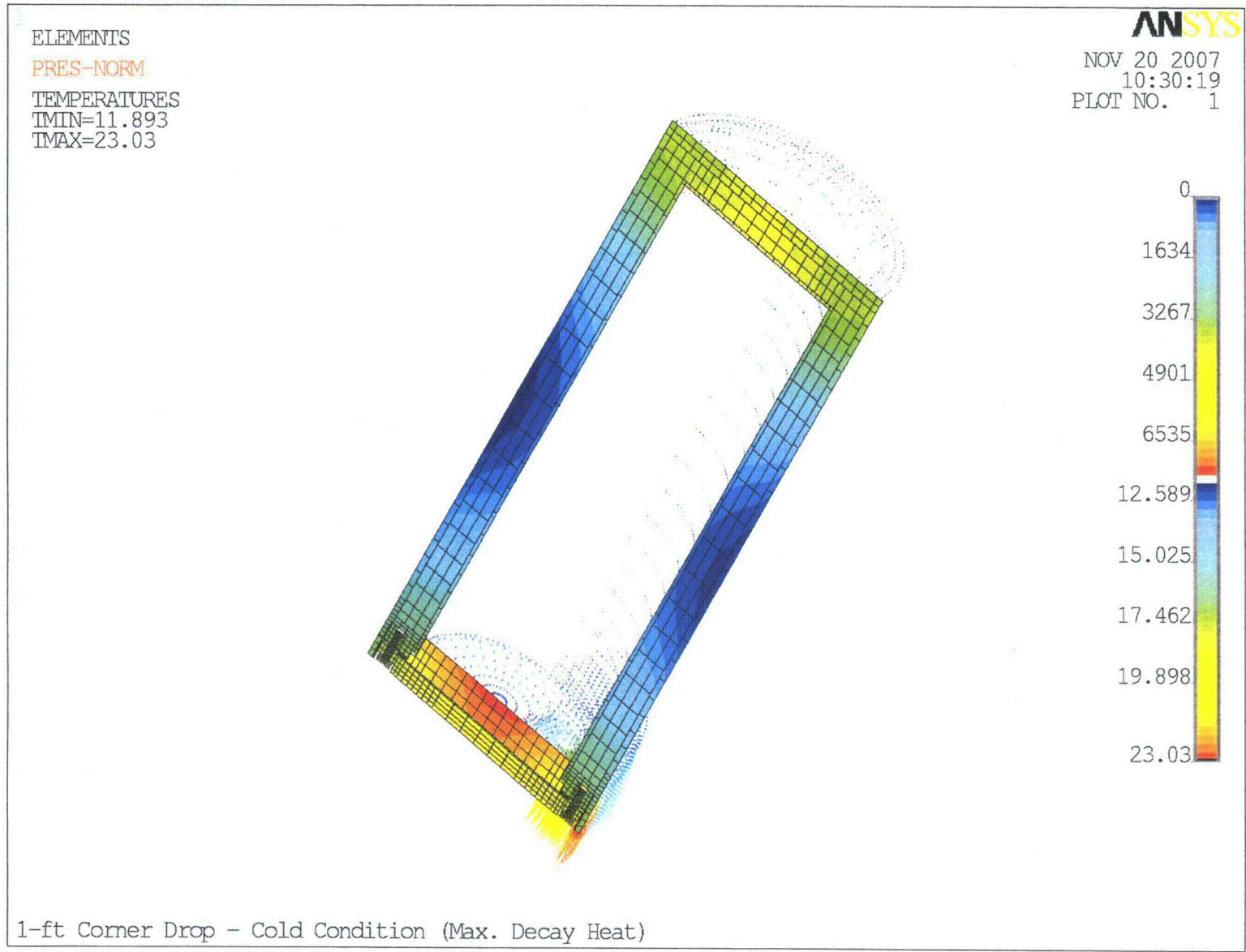
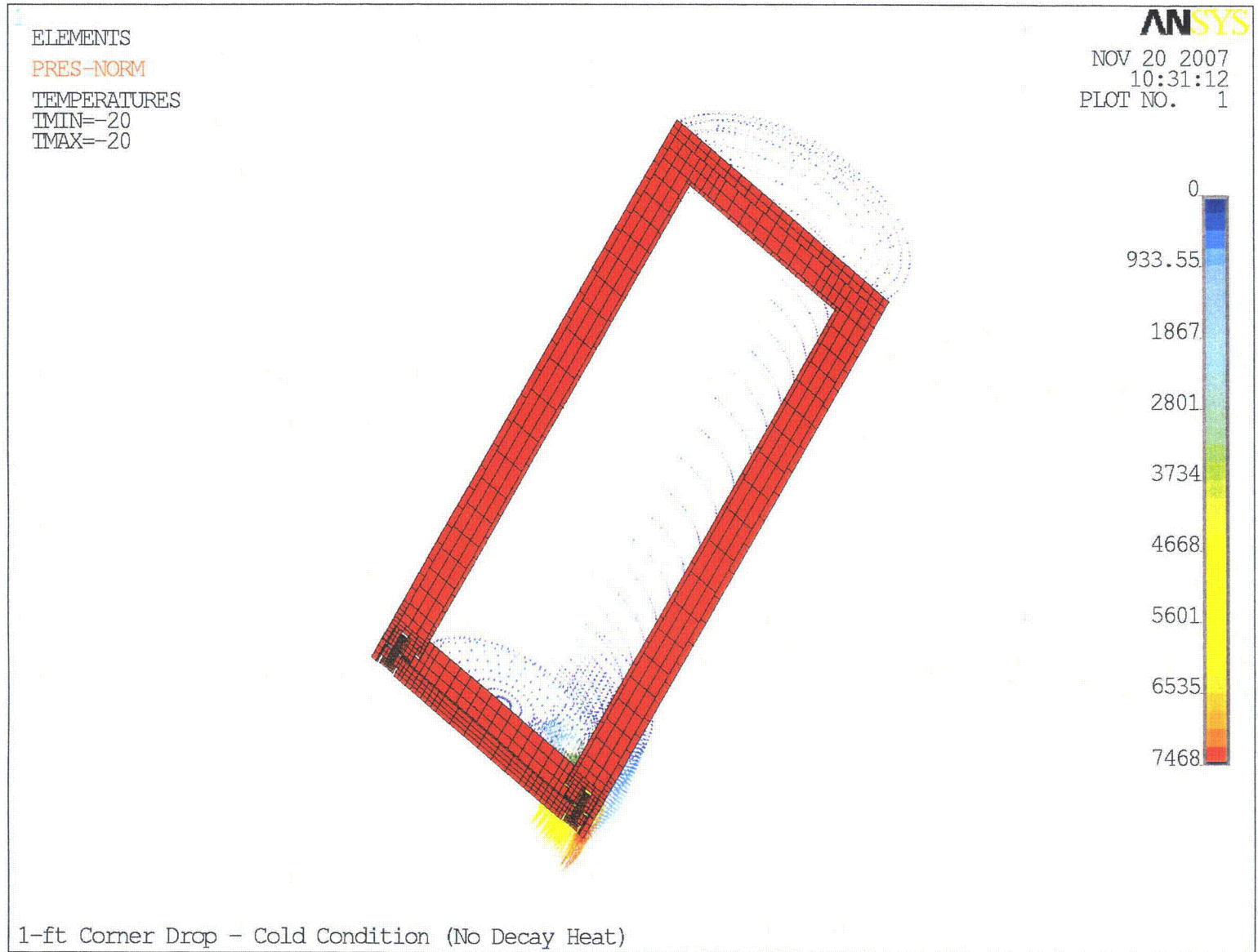


Figure 41

Temperature Profile and Pressure Distribution Used for 1-ft Corner Drop – Load Combination No.2



Title \_\_\_\_\_ Structural Analyses of the 3-60B Cask Under Drop Conditions  
 Calc. No. ST-504 (Figures) \_\_\_\_\_ Rev. 1 \_\_\_\_\_ Sheet 42 of 45

Figure 42  
 Temperature Profile and Pressure Distribution Used for 1-ft Corner Drop – Load Combination No.3



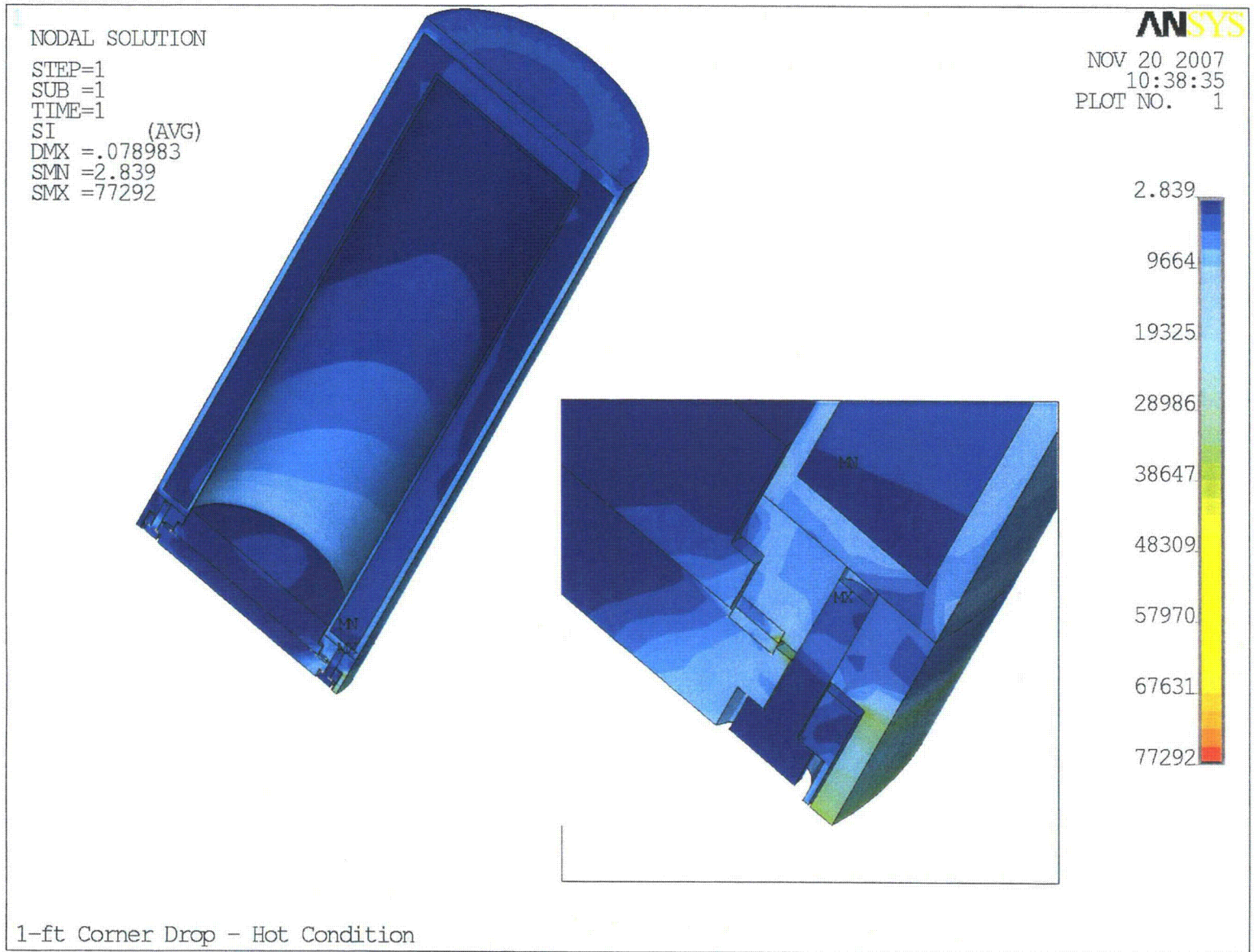


Figure 43  
 Stress Intensity Plot – 1-ft Corner Drop – Load Combination No.1

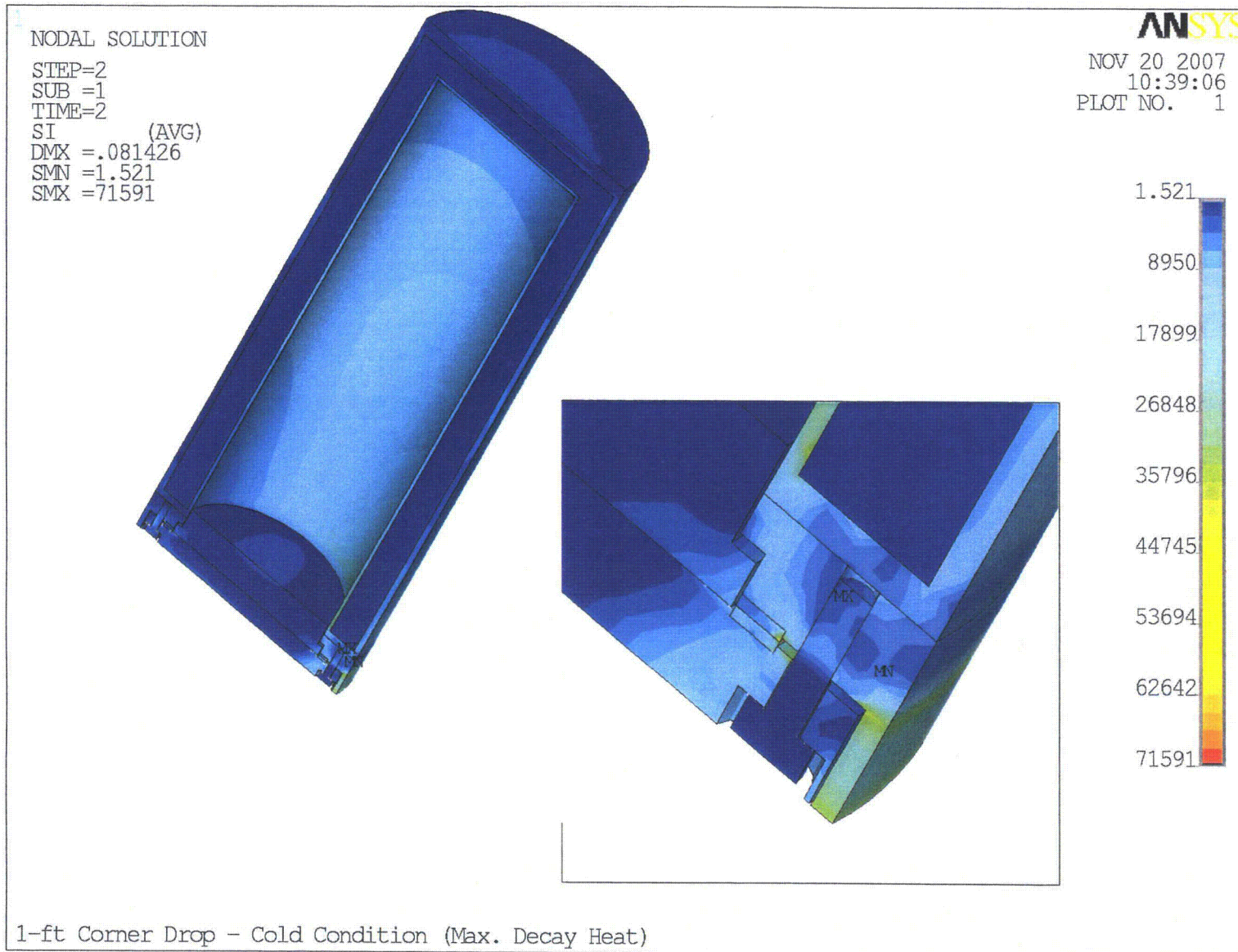


Figure 44

Stress Intensity Plot - 1-ft Corner Drop - Load Combination No.2



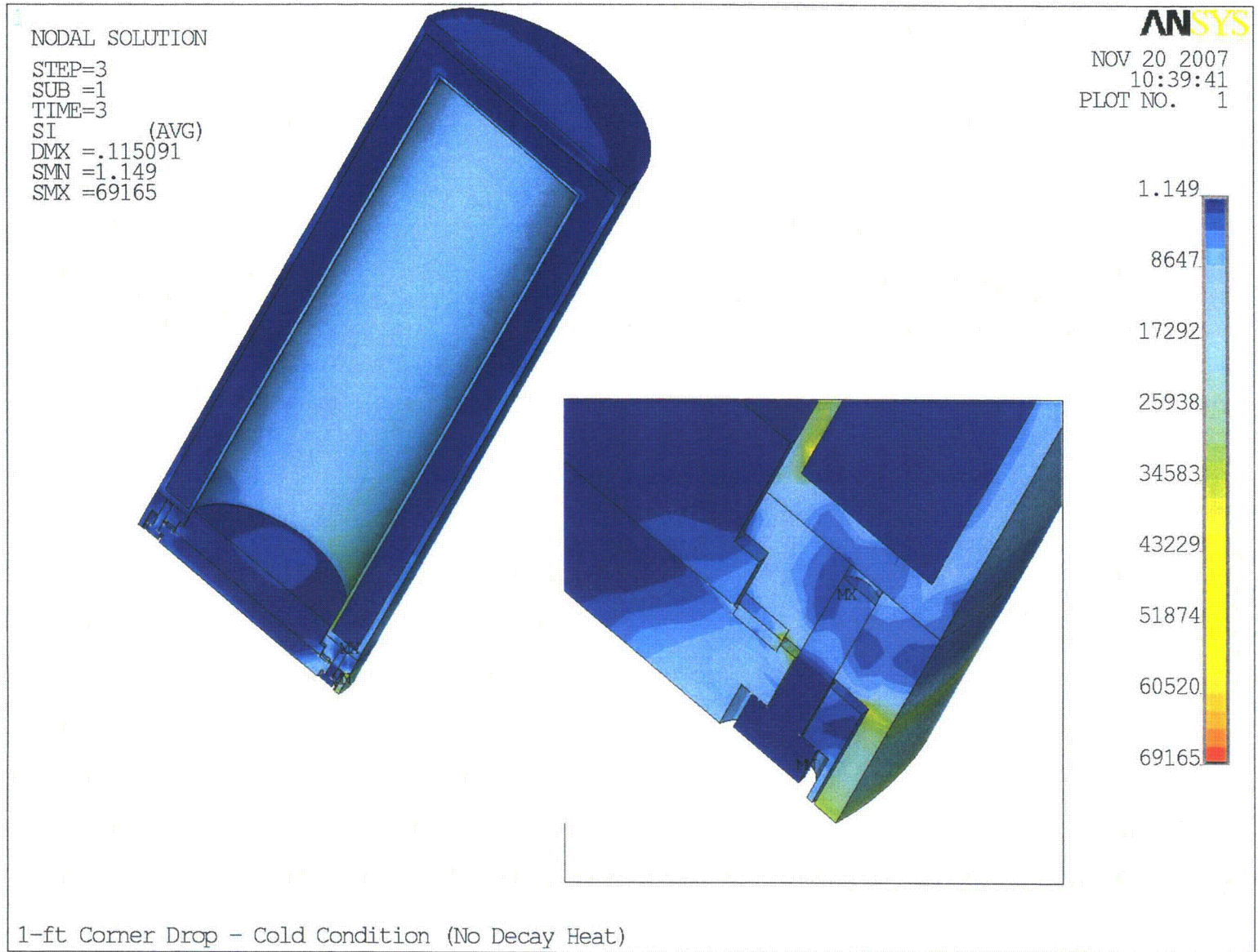


Figure 45  
 Stress Intensity Plot - 1-ft Corner Drop - Load Combination No.3



**Title** Structural Analyses of the 3-60B Cask Under Drop Conditions

**Calc. No.** ST-504 **Rev.** 1

**Sheet** 26 **of** 27

**Appendix 1**

Printout of the ANSYS Model Data

(16 Pages)

# 3-60B Cask Drop Analyses

By Dr. Mirza I. Baig

Report Generated by ANSYS

## Title Listing

\*\*\*\*\* TITLES \*\*\*\*\*

\*\*\* YOU ARE IN ANSYS - ENGINEERING ANALYSIS SYSTEM \*\*\*

ANSYS Mechanical/Emag

RELEASE 11.0SP1 UPDATE 20070830 CUSTOMER 00222442

INITIAL JOBNAME = file

CURRENT JOBNAME = file

Current Working Directory: Y:\30-ft Drop\End

TITLE= 30-ft End Drop - Cold Condition (No Decay Heat)

MENULIST File: C:\Program Files\ANSYS Inc\v110\ANSYS\gui\en-us\UIDL\menulist110.ans

## Global Status

G L O B A L S T A T U S

ANSYS - Engineering Analysis System

Dec 07, 2007

11:27

Release 11.0SP1

00222442

INTEL NT

Version

Current working directory: Y:\30-ft Drop\End

MENULIST File: C:\Program Files\ANSYS Inc\v110\ANSYS\gui\en-us\UIDL\menulist110.ans

Product(s) enabled: ANSYS Mechanical/Emag

Total connect time. . . . . 0 hours 1 minutes

Total CP usage. . . . . 0 hours 0 minutes 4.4 seconds

J O B I N F O R M A T I O N -----

30-ft End Drop - Cold Condition (No Decay Heat)

Current jobname . . . . .file

Initial jobname . . . . .file

Units . . . . . unknown

	Available	Used
Scratch Memory Space. . . . .	256.000 mb	4.919 mb ( 1.9%)
Database space . . . . .	65535.750 mb	129.391 mb ( 0.2%)

User menu file in use . . .C:\Program Files\ANSYS Inc\v110\ANSYS\gui\en-us\uidl\UIMENU.GRN  
 User menu file in use . . .C:\Program Files\ANSYS Inc\v110\ANSYS\gui\en-us\uidl\UIFUNC1.GRN  
 User menu file in use . . .C:\Program Files\ANSYS Inc\v110\ANSYS\gui\en-us\uidl\UIFUNC2.GRN  
 User menu file in use . . .C:\Program Files\ANSYS Inc\v110\ANSYS\gui\en-us\uidl\MECHTOOL.AUI  
 Beta features . . . . .are not shown in the user interface

M O D E L I N F O R M A T I O N -----

Solid model summary:

	Largest Number	Number Defined	Number Selected
Keypoints . . . . .	0	0	0
Lines . . . . .	0	0	0
Areas . . . . .	0	0	0
Volumes . . . . .	0	0	0

Finite element model summary:

	Largest Number	Number Defined	Number Selected
Nodes . . . . .	47887	36999	36999
Elements. . . . .	37659	37659	24352
Element types . . . . .	69	65	n.a.
Real constant sets. . . . .	35	31	n.a.
Material property sets. . . . .	3	3	n.a.
Coupling. . . . .	0	0	n.a.
Constraint equations. . . . .	0	0	n.a.
Master DOFs . . . . .	0	0	n.a.
Dynamic gap conditions. . . . .	0	0	n.a.

B O U N D A R Y C O N D I T I O N I N F O R M A T I O N -----

	Number Defined
Constraints on nodes. . . . .	2851
Constraints on keypoints. . . . .	0
Constraints on lines. . . . .	0
Constraints on areas. . . . .	0
Forces on nodes . . . . .	0
Forces on keypoints . . . . .	0
Surface loads on elements . . . . .	3216
Number of element flagged surfaces . . . . .	0

Surface loads on lines. . . . . 0  
 Surface loads on areas. . . . . 0  
 Body loads on elements. . . . . 0  
 Body loads on nodes . . . . . 0  
 Body loads on keypoints . . . . . 0

Temperatures

Uniform temperature. . . . . -20.000  
 Reference temperature. . . . . 70.000  
 Offset from absolute scale . . . . . 0.000

	X	Y	Z
Linear acceleration . . . . .	0.0000	0.0000	166.00
Angular velocity (about global CS). . . . .	0.0000	0.0000	0.0000
Angular acceleration (about global CS). . . . .	0.0000	0.0000	0.0000
Location of reference CS. . . . .	0.0000	0.0000	0.0000
Angular velocity (about reference CS) . . . . .	0.0000	0.0000	0.0000
Angular acceleration (about reference CS) . . . . .	0.0000	0.0000	0.0000

R O U T I N E I N F O R M A T I O N -----

Current routine. . . . .Preprocessing (PREP7)

Active coordinate system . . . . . 12 (Cartesian)

Display coordinate system. . . . . 0 (Cartesian)

Current element attributes:

Type number . . . . . 69 (CONTA174)  
 Real number . . . . . 35  
 Material number . . . . . 1  
 Element coordinate system number. . . . . 0

Current mesher type. . . . .based on default element shape

Current element meshing shape 2D . . .use default element shape.

Current element meshing shape 3D . . .use default element shape.

SmrtSize Level . . . . . OFF

Global element size. . . . . 0 divisions per line

Active coordinate system . . . . . 12 (Cartesian)

Display coordinate system. . . . . 0 (Cartesian)

Analysis type. . . . .Static (steady-state)

Active options for this analysis type:

Large deformation effects . . . . .Not included  
 Plasticity. . . . .Not included  
 Creep . . . . .Not included  
 Equation solver to use. . . . .Program Chosen

Results file . . . . .file.rst

Load step number . . . . . 4  
 Number of substeps . . . . . 1  
 Step change boundary conditions . .No

---

**Solution Status**

S O L U T I O N   O P T I O N S

PROBLEM DIMENSIONALITY. . . . . 3-D  
 DEGREES OF FREEDOM. . . . . UX    UY    UZ  
 ANALYSIS TYPE . . . . . .STATIC (STEADY-STATE)  
 NEWTON-RAPHSON OPTION . . . . . .PROGRAM CHOSEN  
 GLOBALLY ASSEMBLED MATRIX . . . . . .SYMMETRIC

L O A D   S T E P   O P T I O N S

LOAD STEP NUMBER. . . . . 4  
 TIME AT END OF THE LOAD STEP. . . . . 1.0000  
 NUMBER OF SUBSTEPS. . . . . 1  
 MAXIMUM NUMBER OF EQUILIBRIUM ITERATIONS. . . . . 15  
 STEP CHANGE BOUNDARY CONDITIONS . . . . . NO  
 TERMINATE ANALYSIS IF NOT CONVERGED . . . . .YES (EXIT)  
 CONVERGENCE CONTROLS. . . . . .USE DEFAULTS  
 INERTIA LOADS                            X            Y            Z  
   ACEL . . . . . 0.0000            0.0000            166.00  
 PRINT OUTPUT CONTROLS . . . . . .NO PRINTOUT  
 DATABASE OUTPUT CONTROLS  
   ITEM            FREQUENCY      COMPONENT  
   BASI            ALL

**Element Type Listing**

---

LIST ELEMENT TYPES FROM    1 TO    69 BY    1

ELEMENT TYPE	1 IS SOLID185	3-D 8-NODE STRUCTURAL SOLID	INOPR
KEYOPT(1-12)=	0 0 0 0 0 0	0 0 0 0 0 0 0	0
ELEMENT TYPE	2 IS SHELL41	MEMBRANE SHELL	INOPR
KEYOPT(1-12)=	0 0 0 0 0 0	0 0 0 0 0 0 0	0
ELEMENT TYPE	3 IS SOLSH190	3-D 8-NODE SOLID SHELL	INOPR
KEYOPT(1-12)=	0 0 0 0 0 0	0 0 0 0 0 0 0	0
ELEMENT TYPE	4 IS TARGET170	3-D TARGET SEGMENT	INOPR
KEYOPT(1-12)=	0 0 0 0 0 0	0 0 0 0 0 0 0	0
ELEMENT TYPE	5 IS CONTA175	NODE-TO-SURFACE CONTACT	INOPR
KEYOPT(1-12)=	0 2 0 0 3 0	0 0 1 2 0 5	0

ELEMENT TYPE	6	IS	TARGE170		3-D TARGET SEGMENT		INOPR
KEYOPT(1-12) =	0	0	0	0	0	0	0
ELEMENT TYPE	7	IS	CONTA175		NODE-TO-SURFACE CONTACT		INOPR
KEYOPT(1-12) =	0	0	0	0	3	0	0
ELEMENT TYPE	8	IS	TARGE170		3-D TARGET SEGMENT		INOPR
KEYOPT(1-12) =	0	0	0	0	0	0	0
ELEMENT TYPE	9	IS	CONTA175		NODE-TO-SURFACE CONTACT		INOPR
KEYOPT(1-12) =	0	0	0	0	3	0	0
ELEMENT TYPE	10	IS	TARGE170		3-D TARGET SEGMENT		INOPR
KEYOPT(1-12) =	0	0	0	0	0	0	0
ELEMENT TYPE	11	IS	CONTA175		NODE-TO-SURFACE CONTACT		INOPR
KEYOPT(1-12) =	0	0	0	0	3	0	0
ELEMENT TYPE	12	IS	TARGE170		3-D TARGET SEGMENT		INOPR
KEYOPT(1-12) =	0	0	0	0	0	0	0
ELEMENT TYPE	13	IS	CONTA175		NODE-TO-SURFACE CONTACT		INOPR
KEYOPT(1-12) =	0	0	0	0	3	0	0
ELEMENT TYPE	14	IS	TARGE170		3-D TARGET SEGMENT		INOPR
KEYOPT(1-12) =	0	0	0	0	0	0	0
ELEMENT TYPE	15	IS	CONTA175		NODE-TO-SURFACE CONTACT		INOPR
KEYOPT(1-12) =	0	0	0	0	3	0	0
ELEMENT TYPE	16	IS	TARGE170		3-D TARGET SEGMENT		INOPR
KEYOPT(1-12) =	0	0	0	0	0	0	0
ELEMENT TYPE	17	IS	CONTA175		NODE-TO-SURFACE CONTACT		INOPR
KEYOPT(1-12) =	0	0	0	0	3	0	0
ELEMENT TYPE	18	IS	TARGE170		3-D TARGET SEGMENT		INOPR
KEYOPT(1-12) =	0	0	0	0	0	0	0
ELEMENT TYPE	19	IS	CONTA175		NODE-TO-SURFACE CONTACT		INOPR
KEYOPT(1-12) =	0	0	0	0	3	0	0
ELEMENT TYPE	20	IS	TARGE170		3-D TARGET SEGMENT		INOPR
KEYOPT(1-12) =	0	0	0	0	0	0	0
ELEMENT TYPE	21	IS	CONTA175		NODE-TO-SURFACE CONTACT		INOPR
KEYOPT(1-12) =	0	0	0	0	3	0	0
ELEMENT TYPE	22	IS	TARGE170		3-D TARGET SEGMENT		INOPR
KEYOPT(1-12) =	0	0	0	0	0	0	0
ELEMENT TYPE	23	IS	CONTA175		NODE-TO-SURFACE CONTACT		INOPR
KEYOPT(1-12) =	0	0	0	0	3	0	0
ELEMENT TYPE	24	IS	TARGE170		3-D TARGET SEGMENT		INOPR
KEYOPT(1-12) =	0	0	0	0	0	0	0



ELEMENT TYPE	25	IS	CONTA174		3D 8-NODE SURF-SURF CONTACT	INOPR
KEYOPT(1-12)=	0	0	0	0 3	0 0 0 1 2 0 5	0
ELEMENT TYPE	26	IS	TARGE170		3-D TARGET SEGMENT	INOPR
KEYOPT(1-12)=	0	0	0	0 0	0 0 0 0 0 0 0	0
ELEMENT TYPE	27	IS	CONTA174		3D 8-NODE SURF-SURF CONTACT	INOPR
KEYOPT(1-12)=	0	0	0	0 3	0 0 0 1 2 0 5	0
ELEMENT TYPE	28	IS	TARGE170		3-D TARGET SEGMENT	INOPR
KEYOPT(1-12)=	0	0	0	0 0	0 0 0 0 0 0 0	0
ELEMENT TYPE	29	IS	CONTA174		3D 8-NODE SURF-SURF CONTACT	INOPR
KEYOPT(1-12)=	0	0	0	0 3	0 0 0 1 2 0 5	0
ELEMENT TYPE	30	IS	TARGE170		3-D TARGET SEGMENT	INOPR
KEYOPT(1-12)=	0	0	0	0 0	0 0 0 0 0 0 0	0
ELEMENT TYPE	31	IS	CONTA174		3D 8-NODE SURF-SURF CONTACT	INOPR
KEYOPT(1-12)=	0	0	0	0 3	0 0 0 1 2 0 5	0
ELEMENT TYPE	32	IS	TARGE170		3-D TARGET SEGMENT	INOPR
KEYOPT(1-12)=	0	0	0	0 0	0 0 0 0 0 0 0	0
ELEMENT TYPE	33	IS	CONTA174		3D 8-NODE SURF-SURF CONTACT	INOPR
KEYOPT(1-12)=	0	0	0	0 3	0 0 0 1 2 0 5	0
ELEMENT TYPE	38	IS	TARGE170		3-D TARGET SEGMENT	INOPR
KEYOPT(1-12)=	0	0	0	0 0	0 0 0 0 0 0 0	0
ELEMENT TYPE	39	IS	CONTA174		3D 8-NODE SURF-SURF CONTACT	INOPR
KEYOPT(1-12)=	0	0	0	0 3	0 0 0 1 2 0 5	0
ELEMENT TYPE	40	IS	TARGE170		3-D TARGET SEGMENT	INOPR
KEYOPT(1-12)=	0	0	0	0 0	0 0 0 0 0 0 0	0
ELEMENT TYPE	41	IS	CONTA174		3D 8-NODE SURF-SURF CONTACT	INOPR
KEYOPT(1-12)=	0	0	0	0 3	0 0 0 1 2 0 5	0
ELEMENT TYPE	42	IS	TARGE170		3-D TARGET SEGMENT	INOPR
KEYOPT(1-12)=	0	0	0	0 0	0 0 0 0 0 0 0	0
ELEMENT TYPE	43	IS	CONTA174		3D 8-NODE SURF-SURF CONTACT	INOPR
KEYOPT(1-12)=	0	0	0	0 3	0 0 0 1 2 0 5	0
ELEMENT TYPE	44	IS	TARGE170		3-D TARGET SEGMENT	INOPR
KEYOPT(1-12)=	0	0	0	0 0	0 0 0 0 0 0 0	0
ELEMENT TYPE	45	IS	CONTA174		3D 8-NODE SURF-SURF CONTACT	INOPR
KEYOPT(1-12)=	0	0	0	0 3	0 0 0 1 2 0 5	0
ELEMENT TYPE	46	IS	TARGE170		3-D TARGET SEGMENT	INOPR
KEYOPT(1-12)=	0	0	0	0 0	0 0 0 0 0 0 0	0
ELEMENT TYPE	47	IS	CONTA174		3D 8-NODE SURF-SURF CONTACT	INOPR
KEYOPT(1-12)=	0	2	0	2 3	0 0 0 1 2 0 6	0



```

ELEMENT TYPE      67 IS CONTA174      3D 8-NODE SURF-SURF CONTACT      INOPR
KEYOPT(1-12)=    0 0 0 0 3 0 0 0 1 2 0 0      0

ELEMENT TYPE      68 IS TARGE170      3-D TARGET SEGMENT      INOPR
KEYOPT(1-12)=    0 0 0 0 0 0 0 0 0 0 0 0      0

ELEMENT TYPE      69 IS CONTA174      3D 8-NODE SURF-SURF CONTACT      INOPR
KEYOPT(1-12)=    0 0 0 0 3 0 0 0 1 2 0 0      0

CURRENT NODAL DOF SET IS  UX    UY    UZ
THREE-DIMENSIONAL MODEL

```

---

### Real Constant Listing

```

LIST REAL SETS      1 TO      35 BY      1

REAL CONSTANT SET      3 ITEMS  1 TO  6
  0.0000      0.0000      1.0000      0.10000      0.0000      0.0000

REAL CONSTANT SET      3 ITEMS  7 TO 12
  0.0000      0.0000      0.10000E+21      0.0000      1.0000      0.0000

REAL CONSTANT SET      3 ITEMS 13 TO 18
  0.0000      0.0000      1.0000      0.0000      1.0000      0.50000

REAL CONSTANT SET      3 ITEMS 19 TO 24
  0.0000      1.0000      1.0000      0.0000      0.0000      1.0000

REAL CONSTANT SET      3 ITEMS 25 TO 25
  10.000

REAL CONSTANT SET      4 ITEMS  1 TO  6
  0.0000      0.0000      1.0000      0.10000      0.0000      0.0000

REAL CONSTANT SET      4 ITEMS  7 TO 12
  0.0000      0.0000      0.10000E+21      0.0000      1.0000      0.0000

REAL CONSTANT SET      4 ITEMS 13 TO 18
  0.0000      0.0000      1.0000      0.0000      1.0000      0.50000

REAL CONSTANT SET      4 ITEMS 19 TO 24
  0.0000      1.0000      1.0000      0.0000      0.0000      1.0000

REAL CONSTANT SET      4 ITEMS 25 TO 25
  10.000

REAL CONSTANT SET      5 ITEMS  1 TO  6
  0.0000      0.0000      1.0000      0.10000      0.0000      0.0000

REAL CONSTANT SET      5 ITEMS  7 TO 12
  0.0000      0.0000      0.10000E+21      0.0000      1.0000      0.0000

REAL CONSTANT SET      5 ITEMS 13 TO 18
  0.0000      0.0000      1.0000      0.0000      1.0000      0.50000

```

REAL CONSTANT SET	5	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	5	ITEMS 25 TO 25	10.000					
REAL CONSTANT SET	6	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	6	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	6	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	6	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	6	ITEMS 25 TO 25	10.000					
REAL CONSTANT SET	7	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	7	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	7	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	7	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	7	ITEMS 25 TO 25	10.000					
REAL CONSTANT SET	8	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	8	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	8	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	8	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	8	ITEMS 25 TO 25	10.000					
REAL CONSTANT SET	9	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	9	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000

REAL CONSTANT SET	9	ITEMS 13 TO	18			
0.0000	0.0000	1.0000	0.0000	1.0000	0.50000	
REAL CONSTANT SET	9	ITEMS 19 TO	24			
0.0000	1.0000	1.0000	0.0000	0.0000	1.0000	
REAL CONSTANT SET	9	ITEMS 25 TO	25			
10.000						
REAL CONSTANT SET	10	ITEMS 1 TO	6			
0.0000	0.0000	1.0000	0.10000	0.0000	0.0000	
REAL CONSTANT SET	10	ITEMS 7 TO	12			
0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000	
REAL CONSTANT SET	10	ITEMS 13 TO	18			
0.0000	0.0000	1.0000	0.0000	1.0000	0.50000	
REAL CONSTANT SET	10	ITEMS 19 TO	24			
0.0000	1.0000	1.0000	0.0000	0.0000	1.0000	
REAL CONSTANT SET	10	ITEMS 25 TO	25			
10.000						
REAL CONSTANT SET	11	ITEMS 1 TO	6			
0.0000	0.0000	1.0000	0.10000	0.0000	0.0000	
REAL CONSTANT SET	11	ITEMS 7 TO	12			
0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000	
REAL CONSTANT SET	11	ITEMS 13 TO	18			
0.0000	0.0000	1.0000	0.0000	1.0000	0.50000	
REAL CONSTANT SET	11	ITEMS 19 TO	24			
0.0000	1.0000	1.0000	0.0000	0.0000	1.0000	
REAL CONSTANT SET	11	ITEMS 25 TO	25			
10.000						
REAL CONSTANT SET	12	ITEMS 1 TO	6			
0.0000	0.0000	1.0000	0.10000	0.0000	0.0000	
REAL CONSTANT SET	12	ITEMS 7 TO	12			
0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000	
REAL CONSTANT SET	12	ITEMS 13 TO	18			
0.0000	0.0000	1.0000	0.0000	1.0000	0.50000	
REAL CONSTANT SET	12	ITEMS 19 TO	24			
0.0000	1.0000	1.0000	0.0000	0.0000	1.0000	
REAL CONSTANT SET	12	ITEMS 25 TO	25			
10.000						
REAL CONSTANT SET	13	ITEMS 1 TO	6			
0.0000	0.0000	1.0000	0.10000	0.0000	0.0000	

REAL CONSTANT SET	13	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	13	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	13	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	14	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	14	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	14	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	14	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	15	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	15	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	15	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	15	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	16	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	16	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	16	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	16	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	17	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	17	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	17	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	17	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000



REAL CONSTANT SET	20	ITEMS 1 TO	6			
0.0000	0.0000	1.0000	0.10000	0.0000	0.0000	
REAL CONSTANT SET	20	ITEMS 7 TO	12			
0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000	
REAL CONSTANT SET	20	ITEMS 13 TO	18			
0.0000	0.0000	1.0000	0.0000	1.0000	0.50000	
REAL CONSTANT SET	20	ITEMS 19 TO	24			
0.0000	1.0000	1.0000	0.0000	0.0000	1.0000	
REAL CONSTANT SET	21	ITEMS 1 TO	6			
0.0000	0.0000	1.0000	0.10000	0.0000	0.0000	
REAL CONSTANT SET	21	ITEMS 7 TO	12			
0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000	
REAL CONSTANT SET	21	ITEMS 13 TO	18			
0.0000	0.0000	1.0000	0.0000	1.0000	0.50000	
REAL CONSTANT SET	21	ITEMS 19 TO	24			
0.0000	1.0000	1.0000	0.0000	0.0000	1.0000	
REAL CONSTANT SET	22	ITEMS 1 TO	6			
0.0000	0.0000	1.0000	0.10000	0.0000	0.0000	
REAL CONSTANT SET	22	ITEMS 7 TO	12			
0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000	
REAL CONSTANT SET	22	ITEMS 13 TO	18			
0.0000	0.0000	1.0000	0.0000	1.0000	0.50000	
REAL CONSTANT SET	22	ITEMS 19 TO	24			
0.0000	1.0000	1.0000	0.0000	0.0000	1.0000	
REAL CONSTANT SET	23	ITEMS 1 TO	6			
0.0000	0.0000	1.0000	0.10000	0.0000	0.0000	
REAL CONSTANT SET	23	ITEMS 7 TO	12			
0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000	
REAL CONSTANT SET	23	ITEMS 13 TO	18			
0.0000	0.0000	1.0000	0.0000	1.0000	0.50000	
REAL CONSTANT SET	23	ITEMS 19 TO	24			
0.0000	1.0000	1.0000	0.0000	0.0000	1.0000	
REAL CONSTANT SET	24	ITEMS 1 TO	6			
0.0000	0.0000	1.0000	0.10000	0.0000	0.0000	
REAL CONSTANT SET	24	ITEMS 7 TO	12			
0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000	
REAL CONSTANT SET	24	ITEMS 13 TO	18			
0.0000	0.0000	1.0000	0.0000	1.0000	0.50000	

REAL CONSTANT SET	24	ITEMS 19 TO	24			
0.0000	1.0000	1.0000	0.0000	0.0000	1.0000	
REAL CONSTANT SET	24	ITEMS 25 TO	30			
10.000	0.0000	0.0000	0.0000	0.0000	0.0000	
REAL CONSTANT SET	25	ITEMS 1 TO	6			
0.0000	0.0000	1.0000	0.10000	0.0000	0.0000	
REAL CONSTANT SET	25	ITEMS 7 TO	12			
0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000	
REAL CONSTANT SET	25	ITEMS 13 TO	18			
0.0000	0.0000	1.0000	0.0000	1.0000	0.50000	
REAL CONSTANT SET	25	ITEMS 19 TO	24			
0.0000	1.0000	1.0000	0.0000	0.0000	1.0000	
REAL CONSTANT SET	26	ITEMS 1 TO	6			
0.0000	0.0000	1.0000	0.10000	0.0000	0.0000	
REAL CONSTANT SET	26	ITEMS 7 TO	12			
0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000	
REAL CONSTANT SET	26	ITEMS 13 TO	18			
0.0000	0.0000	1.0000	0.0000	1.0000	0.50000	
REAL CONSTANT SET	26	ITEMS 19 TO	24			
0.0000	1.0000	1.0000	0.0000	0.0000	1.0000	
REAL CONSTANT SET	26	ITEMS 25 TO	30			
10.000	0.0000	0.0000	0.0000	0.0000	0.0000	
REAL CONSTANT SET	27	ITEMS 1 TO	6			
0.0000	0.0000	1.0000	0.10000	0.0000	0.0000	
REAL CONSTANT SET	27	ITEMS 7 TO	12			
0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000	
REAL CONSTANT SET	27	ITEMS 13 TO	18			
0.0000	0.0000	1.0000	0.0000	1.0000	0.50000	
REAL CONSTANT SET	27	ITEMS 19 TO	24			
0.0000	1.0000	1.0000	0.0000	0.0000	1.0000	
REAL CONSTANT SET	27	ITEMS 25 TO	30			
10.000	0.0000	0.0000	0.0000	0.0000	0.0000	
REAL CONSTANT SET	28	ITEMS 1 TO	6			
0.0000	0.0000	1.0000	0.10000	0.0000	0.0000	
REAL CONSTANT SET	28	ITEMS 7 TO	12			
0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000	
REAL CONSTANT SET	28	ITEMS 13 TO	18			
0.0000	0.0000	1.0000	0.0000	1.0000	0.50000	

REAL CONSTANT SET	28	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	29	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	29	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	29	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	29	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	29	ITEMS 25 TO 30	10.000	0.0000	0.0000	0.0000	0.0000	0.0000
REAL CONSTANT SET	30	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	30	ITEMS 7 TO 12	0.0000	0.0000	17320.	0.0000	1.0000	0.0000
REAL CONSTANT SET	30	ITEMS 13 TO 18	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
REAL CONSTANT SET	30	ITEMS 19 TO 24	0.0000	0.0000	1.0000	0.0000	0.0000	0.0000
REAL CONSTANT SET	31	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	31	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	31	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	31	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	32	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	32	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	32	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	32	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	33	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000

REAL CONSTANT SET	33	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	33	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	33	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	34	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	34	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	34	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	34	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000
REAL CONSTANT SET	35	ITEMS 1 TO 6	0.0000	0.0000	1.0000	0.10000	0.0000	0.0000
REAL CONSTANT SET	35	ITEMS 7 TO 12	0.0000	0.0000	0.10000E+21	0.0000	1.0000	0.0000
REAL CONSTANT SET	35	ITEMS 13 TO 18	0.0000	0.0000	1.0000	0.0000	1.0000	0.50000
REAL CONSTANT SET	35	ITEMS 19 TO 24	0.0000	1.0000	1.0000	0.0000	0.0000	1.0000

### Material Properties Listing

LIST MATERIALS 1 TO 3 BY 1  
PROPERTY= ALL

PROPERTY TABLE EX MAT= 1 NUM. POINTS= 6  
 TEMPERATURE DATA TEMPERATURE DATA TEMPERATURE DATA  
 70.000 0.28300E+08 100.00 0.28100E+08 200.00 0.27600E+08  
 300.00 0.27000E+08 400.00 0.26500E+08 500.00 0.25800E+08

PROPERTY TABLE NUXY MAT= 1 NUM. POINTS= 6  
 TEMPERATURE DATA TEMPERATURE DATA TEMPERATURE DATA  
 70.000 0.30000 100.00 0.30000 200.00 0.30000  
 300.00 0.30000 400.00 0.30000 500.00 0.30000

PROPERTY TABLE ALPX MAT= 1 NUM. POINTS= 6 REFERENCE TEMP. = 70.00  
 TEMPERATURE DATA TEMPERATURE DATA TEMPERATURE DATA  
 70.000 0.85000E-05 100.00 0.86000E-05 200.00 0.89000E-05  
 300.00 0.92000E-05 400.00 0.95000E-05 500.00 0.97000E-05





**Title** Structural Analyses of the 3-60B Cask Under Drop Conditions

**Calc. No.** ST-504 **Rev.** 1

**Sheet** 27 **of** 27

**Appendix 2**

Electronic Data on DVD

(1 DVD)