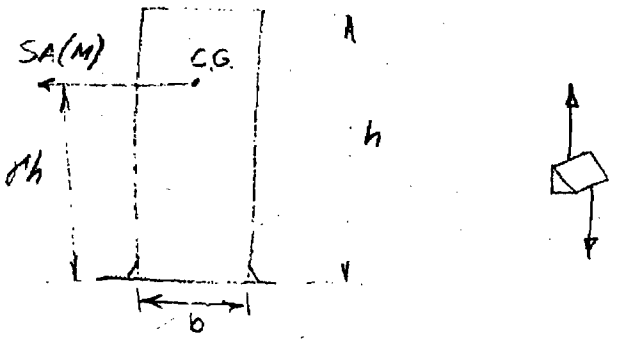
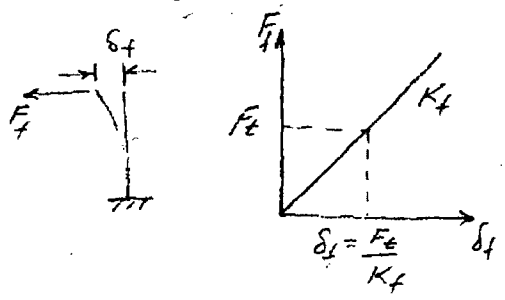
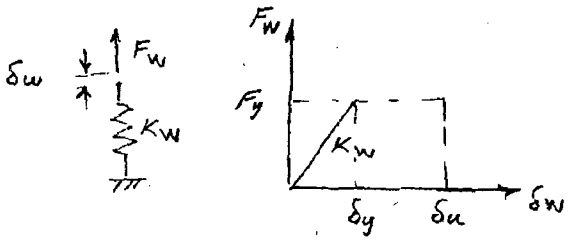
Overturning Resistance Model



Transverse loading of weld support attachment

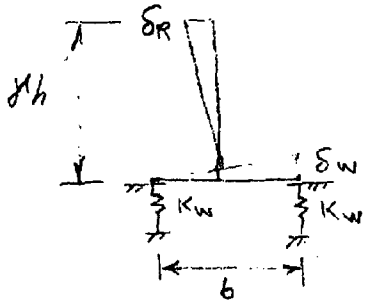


Equipment Resistance Function (linear)



Anchorage Resistance Function (non-linear)

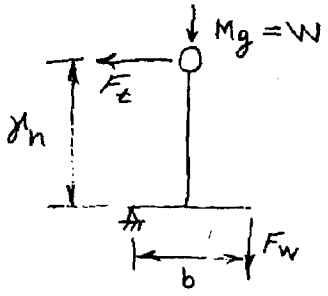
$\delta_u / \delta_y = \mu$ - assume non-linear behavior of support attachment only



Overturning Kinematics

Let $e = \frac{\gamma h}{b}$

$$\delta_R = (\gamma h)\frac{\delta_w}{b} = e\,\delta_w$$

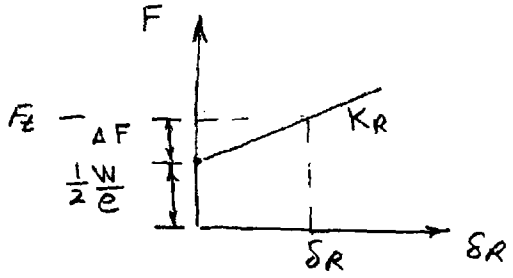


Overturning Mechanics

$$F_t(\gamma h) = F_w b + Mg\frac{b}{2}$$

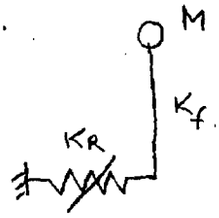
$$F_t = \frac{F_w}{e} + \frac{Mg}{2e}$$

$$\Delta F = F_t - \frac{Mg}{2e} = \frac{F_w}{e}$$



$$\Delta F = K_R\,\delta_R = K_R e\,\delta_w = \frac{K_w\,\delta_w}{e}$$

$$K_R = K_w / e^2$$

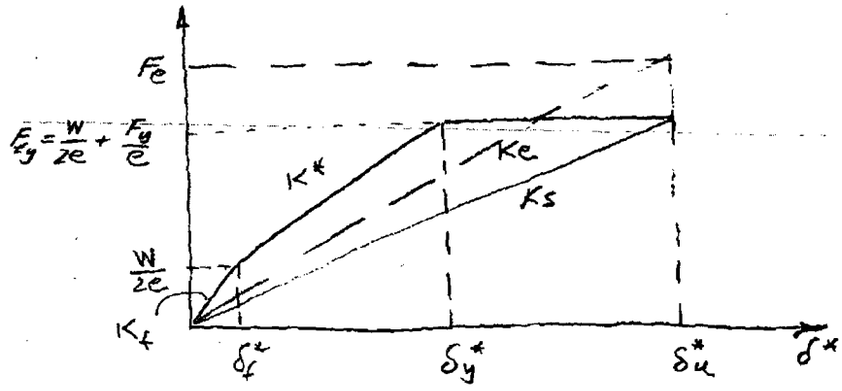


Series Stiffness

$$\frac{1}{K^*} = \frac{1}{K_f} + \frac{1}{K_R}$$

$$K^* = \frac{K_f K_R}{K_f + K_R} = K_f \frac{1}{1 + K_f/K_R}$$

Series Resistance Function (1/4 cycle)



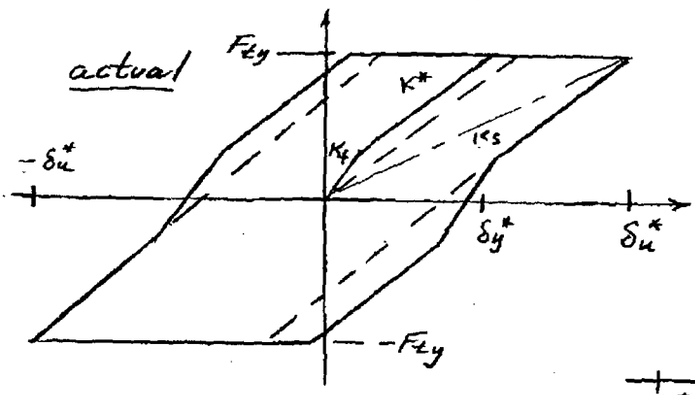
Secant Stiffness

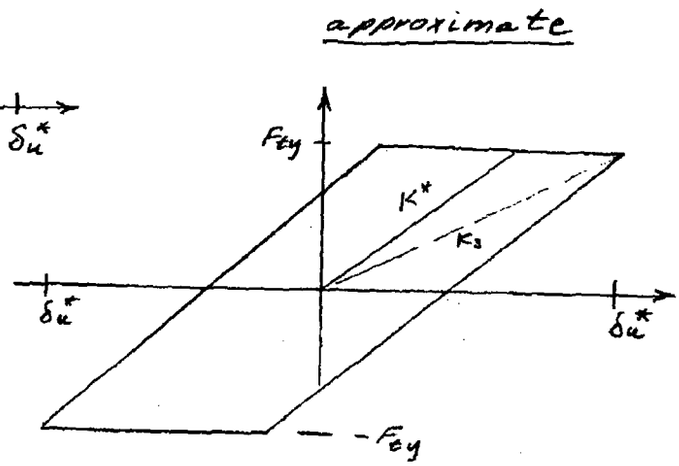
$$K_s = \frac{F_{ty}}{F_{ty}/K_f + e\,\delta_u}$$

$$\delta_f^* = \frac{W}{2e}\frac{1}{K_f} \qquad \delta_y^* = F_{ty}/K_f + e\,\delta_y$$

$$\delta_u^* = F_{ty}/K_f + e\,\delta_u$$

Full Reverse Cycle Hysteretic Loop





– as a conservative approximation, ignore the effect of dead weight restoring force on hysteresis

Define:

$$f_s = \frac{1}{2\pi}\sqrt{K_s/M}$$

$$f_f = \frac{1}{2\pi}\sqrt{K_f/M}\ , \quad K_f = (2\pi f_f)^2 M$$

Let

$$X = \frac{K_s}{K_f} = \frac{f_s^2}{f_f^2} = \frac{F_{ty}}{F_{ty}/K_f + e\,\delta u} = \frac{1}{1 + \dfrac{e\,\delta u\, K_f}{F_{ty}}}$$

$$= \frac{1}{1 + \dfrac{e(2\pi f_f)^2\,\delta u}{F_{ty}/M}}$$

Let $\quad F_{ty}/M = F_{sm}\, SA_r\ , \quad SA_r < SA(f_f, \beta_f)$

$$X = \frac{1}{1 + \dfrac{(2\pi f_f)^2\,\delta u}{F_{sm}/e\ SA_r}}$$

if $\quad SA_c = \dfrac{F_{sm}}{e}\, SA_r$

$$\boxed{X = \frac{1}{1 + \dfrac{(2\pi f_f)^2\,\delta u}{SA_c}}}$$

Given that the definition for X is taken as above and the hysteresis is approximated as indicated, then the same reduction procedure used for the shear resistance model applies, except that A is determined with $SA_c = \frac{F_{sm}}{e} SA_r$. The remainder of the procedure remains the same.