



International Agreement Report

Assessment Study of RELAP-5 MOD-2 Cycle 36.01: Based on the Doel-2 Steam Generator Tube Rupture Incident of June 1979

Prepared by
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Office of Nuclear Regulatory Research
U.S. Nuclear Regulatory Commission
Washington, DC 20555

October 1986

Prepared as part of
The Agreement on Research Participation and Technical Exchange
under the International Thermal-Hydraulic Code Assessment
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EXECUTIVE SUMMARY

Severe plant transients following a Steam Generator Tube Rupture (SGTR) have a relative high probability of occurrence. Such transients provide a direct release path of contaminated coolant to the environment thereby breaching several barriers built in the concept, and induce a rather complex asymmetric plant behaviour.

Analyses of most of the reported SGTR events clearly show that there is substantial operator involvement required to mitigate the consequences of such events for which the operator could rarely follow prescribed procedures. Therefore, the operators had to rely on their skill and insight to bring the plant to a cold shutdown.

All the reported events of this type clearly demonstrate :

- . a need to have better emergency response guidelines;
- . a need to have a better manual control over safety systems such as the high pressure safety injection;
- . a need to better understand the different phenomena which may determine the sequence of events.

The SGTR accident that occurred at the DOEL-2 plant on June 25th 1979 clearly demonstrated above features and triggered a need to acquire the tools and develop the needed skill to cope with these problems.

The resulting data and recordings made during this transient provide a valuable data base for assessing both code and code users, as real plant data are scarcely available and eliminate the scaling problems inherent in small scale test facility data for code assessment.

The code assessment work presented in this document was based on all the available data (from plant computer and plant recordings) from this incident. Although the initial plant conditions were well known, it must be emphasized that the quantity and the quality of the available data from real plants are by nature inferior to well instrumented test facility data. Furthermore, the timing and the intensity of the operator involvement during such transient were not readily available and had to be inferred from the available data.

The objective of this study was to assess the code RELAP-5 MOD-2 CYCLE 36.01 performance, preferentially based on the comparison of trends rather than comparison of absolute values.

The study revealed some important features of the code which can be summarised as follows :

1. The code is capable to simulate the observed phenomena reasonably well and can be used to predict similar transients. However, lack of precise boundary conditions does not allow one to quantify the error margin from this analysis.
2. The code performance is acceptable. The time step is constant and Courant limited for 99 % of the transient, and the execution time on a CYBER-176 is about 1.8 times the transient time. The mass error ratio remains negligible.
3. Impressive improvements over RELAP-5 MOD-1 CYCLE 19 have been observed in the steady break flow rate, and the water vapour flow slip (CCFL) in the pressuriser when pressuriser spray is actuated.
4. Excessive water level swell observed in the intact S.G. during cooldown may be attributed to excessive interphase momentum transfer in the S.G. riser region when bulk boiling is initiated.

5. Excessive interphase mass and heat transfer for condensation and evaporation in quasi stagnant flow conditions was observed in the vapour space of the isolated affected steam generator.
6. Erratic natural circulation in the affected steam generator leads to an anomalous cooldown capacity of the steam generator. This behaviour can be suppressed by inserting high form loss factors in the natural circulation flow path.
7. While some anomalies may be reduced by a judicious choice of the nodalisation and/or code options, this study suggests that further improvements in the constitutive equation package, in function of the various flow regimes is needed to take full benefit of the potential of a 6 equation code.

ABSTRACT

This report presents a code assessment study based on a real plant transient that occurred at the DOEL 2 power plant in Belgium on June 25th 1979.

DOEL 2 is a two-loop WESTINGHOUSE PWR plant of 392 Mwe. A steam generator tube rupture occurred at the end of a heat-up phase which initiated a plant transient which required substantial operator involvement and presented many plant phenomena which are of interest for code assessment.

While real plant transients are of special importance for code validation because of the elimination of code scaling uncertainties, they introduce however some uncertainties related to the specifications of the exact initial and boundary conditions which must be reconstructed from available on-line plant recordings and on-line computer diagnostics.

Best estimate data have been reconstructed for an assessment study by means of the code RELAP5/MOD2/CYCLE 36.01. Because of inherent uncertainties in the plant data, the assessment work is focussed on phenomena whereby the comparison between plant data and computer data is based more on trends than on absolute values. Such approach is able to uncover basic code weaknesses and strengths which can contribute to a better understanding of the code potential.

This work was performed by TRACTIONEL, being the Architect-Engineer for all DOEL plants, in cooperation with the utility EBES which owns and runs these plants.



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1. INTRODUCTION

Severe plant transients following a Steam Generator Tube Rupture (SGTR) have a relative high probability of occurrence (ref. 1). Such transient provide a direct release path of contaminated coolant to the environment thereby breaching several barriers built in the concept, and induce a rather complex asymmetric plant behaviour.

Analyses of most of the reported SGTR events clearly show that there is substantial operator involvement required to mitigate the consequences of such events for which the operator could rarely follow prescribed procedures. Therefore the operators had to rely on their skill and insight to bring the plant to a cold shutdown.

All the reported events of this type clearly demonstrate :

- . a need to have better emergency response guidelines;
- . a need to have a better manual control over safety systems such as the high pressure safety injection;
- . a need to better understand the different phenomena which may determine the sequence of events.

The SGTR accident that occurred at the DOEL-2 plant on June 25th 1979 clearly demonstrated above features and triggered a need to acquire the tools and develop the needed skill to cope with these problems.

The resulting data and recordings made during this transient provide a valuable data base for assessing both code and code users, as real plant data are scarcely available and eliminate the scaling problems inherent in small scale test facility data for code assessment.

The code assessment work presented in this document is based on all the available data (from plant computer and plant recordings) from this incident. Although the initial plant conditions are well known, it must be emphasized that the

quantity and the quality of the available data from real plants are by nature inferior to well instrumented test facility data. Furthermore, the timing and the intensity of the operator involvement during such transient are not readily available and have to be inferred from the available data.

Hence the objective of this study is to assess a code performance preferentially based on the comparison of trends rather than comparison of absolute values.

After a brief description of the plant characteristics (chapter 2) and the anatomy of the SGTR transient (chapter 3), the plant nodalisation for the code RELAP-5 MOD-2 is discussed in chapter 4.

The base case results are described in chapter 5 followed by a discussion of these base case results in chapter 6, and of the sensitivity results in chapter 7. Chapter 8 briefly presents some run statistics.

2. DOEL-2 PLANT AND SYSTEM DESCRIPTION

This section provides a description of the plant and principal systems relevant to understanding the DOEL-2 steam generator tube rupture event during the first hour following the tube rupture. Principal plant components and systems associated with the event include the reactor coolant system, the high pressure safety injection system, portions of the charging and letdown system, the secondary side steam and auxiliary feedwater systems including the power operated atmospheric steam dump valves.

DOEL-2 is a WESTINGHOUSE 2 loop pressurized water reactor (PWR) rated at 392 MWe (NET) and commissioned in 1975, for which TRACTIONEL was the architect/engineer. This plant is part of the twin concept with DOEL-1, as they share some common engineered safety system such as the high pressure safety injection system (HPSI).

2.1. Reactor coolant system

A flow schematic of the reactor coolant system (RCS) is presented in fig. 2.1. The reactor vessel (2R1) is equipped with 2 reactor coolant loops (A and B) comprising each 1 steam generator (2E1A, 2E1B) and 1 main coolant pump (2P1A, 2P1B).

DOEL-2 is equipped with a 8ft core consisting of 121 fuel assemblies with 179 fuel rods per assembly (outside cladding diameter is 10.79 mm). The nominal core power is 1187 MWth. The primary coolant pumps, with a rated power of 2.44 MW each maintain a loop flowrate of 3628 kg/s with a net pump head of 4.275 bar.

A 24 m³ pressuriser (2R2) is connected by a 10" surge line to the "B" loop hot leg.

Two pressuriser spray lines connect the pressuriser spray nozzle with both cold legs, yielding a maximum total spray

flow of 1500 l/min under nominal conditions when spray valves are fully open (2PR001, 2PR002 and 2PR006).

The pressuriser is equipped with pressuriser heaters (proportional + back-up) with a total maximum power of 850 kW. The heaters are switched off automatically when the pressuriser level drops below 20 % .

2.2. High pressure safety injection (SI)

The Doel 1-2 plants are equipped with a common HPSI system⁻ consisting of 4 motor-driven pumps, which through a system of valves and interconnected piping can feed cold water from the refueling water storage tanks into both cold legs of the RCS and also directly into the downcomer of the reactor vessel.

Fig. 2.2 shows a flow schematic of the HPSI system.

Fig. 2.3 and 2.4 illustrate the flow delivery curves of the HPSI system in function of the reactor backpressure for the cases with and without DOWNCOMER injection respectively.

Fig. 2.5. gives the pump characteristic of the HPSI pumps.

The HPSI pumps are actuated on generation of a SI signal. For this event, the SI signal was generated at low - low pressuriser pressure (117 bar) and pumps start feeding cold water when reactor pressure drops below the high head cut-off pressure of the HPSI pumps (108 bar).

2.3. Doel 2 charging and letdown system

The water inventory and primary water chemistry in the reactor coolant system is maintained by the charging and letdown system.

For the charging system, 3 positive displacement pumps take suction from the volumetric control tank and feed cold borated

water via a regenerative heat exchanger into the cold leg of loop B.

The flowrate is normally controlled by the programmed water level in the pressuriser which controls the speed of the pumps. Normally only 1 pump is in operation. The flowrate per pump varies between 8.7 m³/hr for lowest speed of 140 RPM to 14.7 m³/hr at 234 RPM.

The charging system also provides the seal injection water for both main coolant pumps. Part of the seal water goes into the RCS and part returns to the charging system.

The letdown flow is controlled mainly by three orifices in a letdown line connected to the intermediate leg between S.G. and primary pump of loop B. The orifices are sized to discharge respectively 10, 10 and 20 t/hr. The backpressure is controlled by an automatic control valve.

The letdown is closed when the pressuriser water level decreases below 20 % .

Upon generation of a SI signal, phase A containment isolation is initiated which cuts off the compressed - air supply in the reactor building. The containment isolation valve in the letdown line, being a fail close valve, remains closed as long as the SI signal is not RESET.

2.4. Steam generators - Main steam lines - Auxiliary feedwater system

A schematic drawing of the Doel-2 main steam and auxiliary feedwater system is given in fig. 2.6.

The Doel-2 steam generators are U-type Westinghouse series 44 models, with a nominal heat transfer area of 4130 m² consisting of 3260 tubes ($\phi_{out} = 22.2$ m.m) with average length of 18.16 m.

The power-operated atmospheric steam dump valves are hydraulic operated valves each with a steam capacity of 5 % of the total plant steam flow (or 33.3 kg/s at 72 bar).

These valves (one per S.G.) are used to relieve minor pressure increases in the S.G., to avoid lifting the safety valves (6 per SG) and provide also a means of plant cooldown when the condenser is unavailable.

The main steam isolation valves (MSIV) are HOPKINSON valves. During the DOEL 2 event, both MSIV were closed during the heat-up phase and accident.

Also the main feedwater pumps were not operational at the moment the break occurred. The auxiliary feedwater system supplies cold feedwater to the S.G. when the main feedwater system is unavailable.

The system consists of 2 motor-driven auxiliary feedwater pumps (MPA and MPB) and one steam turbine-driven auxiliary pump (TP) (Fig. 2.6.). These pumps take suction from the auxiliary feedwater storage tank and suppletion water storage tank.

The auxiliary feedwater actuation logic is shown in fig. 2.7. The pump characteristics are shown in fig. 2.8. for one motor driven pump, and fig. 2.9 for the steam turbine driven auxiliary feedwater pump.

2.5. Plant diagnostics and uncertainty bands.

2.5.1. Measured plant parameters

Fig. 1 and 2 of annexe 1 illustrate the recorded temperatures

- hot leg loop A : sensor TR/2RC5 (see fig. 2.1.)
- hot leg loop B : sensor TR/2RC25
- cold leg loop A : sensor TR/2RC09
- cold leg loop B : sensor TR/2RC29

The precision of the temperature sensors is estimated at about 1.5 %

- . Fig. 3 and 4 of annexe 1 illustrate the recorder pressures
 - RCS pressure : sensor PRA/2RC11 in hot leg A.
 - pressuriser pressure : sensor PICA/2PR61 in pressuriser.

The precision of the pressure sensors is estimated at ± 2 bar at the nominal system pressure resulting in an uncertainty of 1.3 %

- . Pressuriser level (fig. 5 of annexe 1) : sensor LRCA/2PR11.
The uncertainty band for the level gauge is estimated at 5 %

- . Steam generator pressure

- intact SG (fig. 6 in annexe 1) : sensor MS4A
- affected SG (fig. 7 in annexe 1) : sensor MS4B. These sensors are located just upstream of the main steam isolation valves.

The tolerance on the S.G. pressure gauges is estimated at 3%

- . Steam generator water level (narrow gauge)

- . intact SG (fig. 6 in annexe 1) : sensor FW9A
- . affected SG (fig. 7 in annexe 1) : sensor FW9B.

The tolerance on the S.G. level gauges is estimated at 16 %.

2.5.2. Recorder uncertainties

For the parameter values, a global recorder uncertainty of 3 % should be added to the instrument uncertainty.

For the timing, an estimated off-set of 2 min should be accounted for, from comparing time values in the computer listing to recorded times on figures 1 to 6 of annexe 1.

Furthermore, by comparing fig. 1 and fig. 2 of annexe 1, a large horizontal time shift (about 20 min) is evident due to

improper adjustement of timing. Hence, some engineering judgment is required to synchronise the recorder data which can be done on the basis of some important events (time of break) and on the basis of the timing listed in the plant computer listing.

1. The first part of the document discusses the importance of maintaining accurate records of all transactions and activities. It emphasizes the need for transparency and accountability in financial reporting.

2. The second part of the document outlines the various methods and techniques used to collect and analyze data. It includes a detailed description of the experimental procedures and the statistical analysis performed.

3. The third part of the document presents the results of the study, including a comparison of the different methods and techniques used. It discusses the implications of the findings and provides recommendations for future research.

FIG 2.2 - FLOW DIAGRAM OF HIGH PRESSURE SAFETY INJECTION SYSTEM

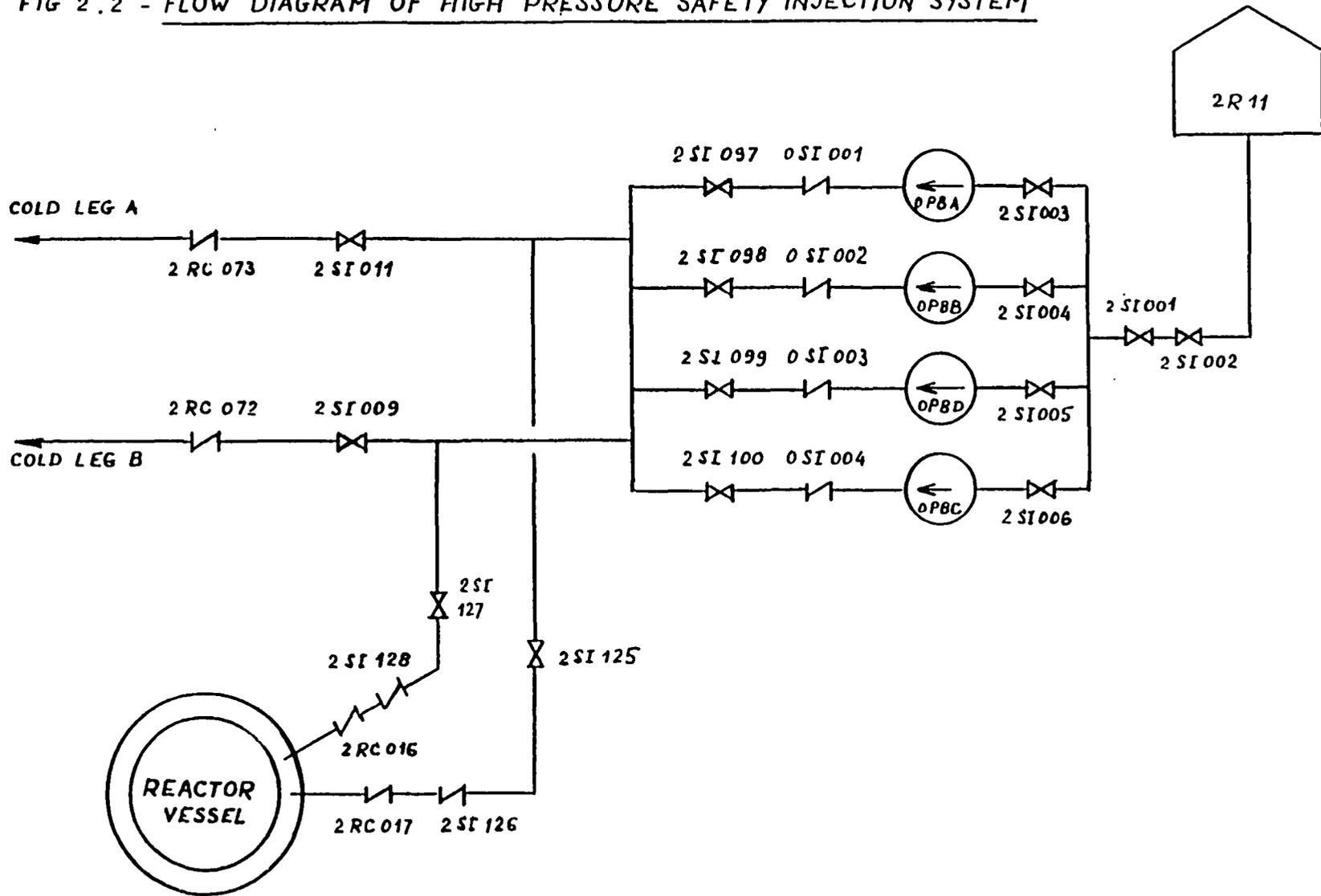


FIG: 2.3 - DOEL 2 HPSI ECCS FLOW DELIVERY CURVES FOR FOUR LINES CONNECTING TO BOTH CL AND HL OR TO BOTH COLD LEGS AND VESSEL DOWNCOMER.

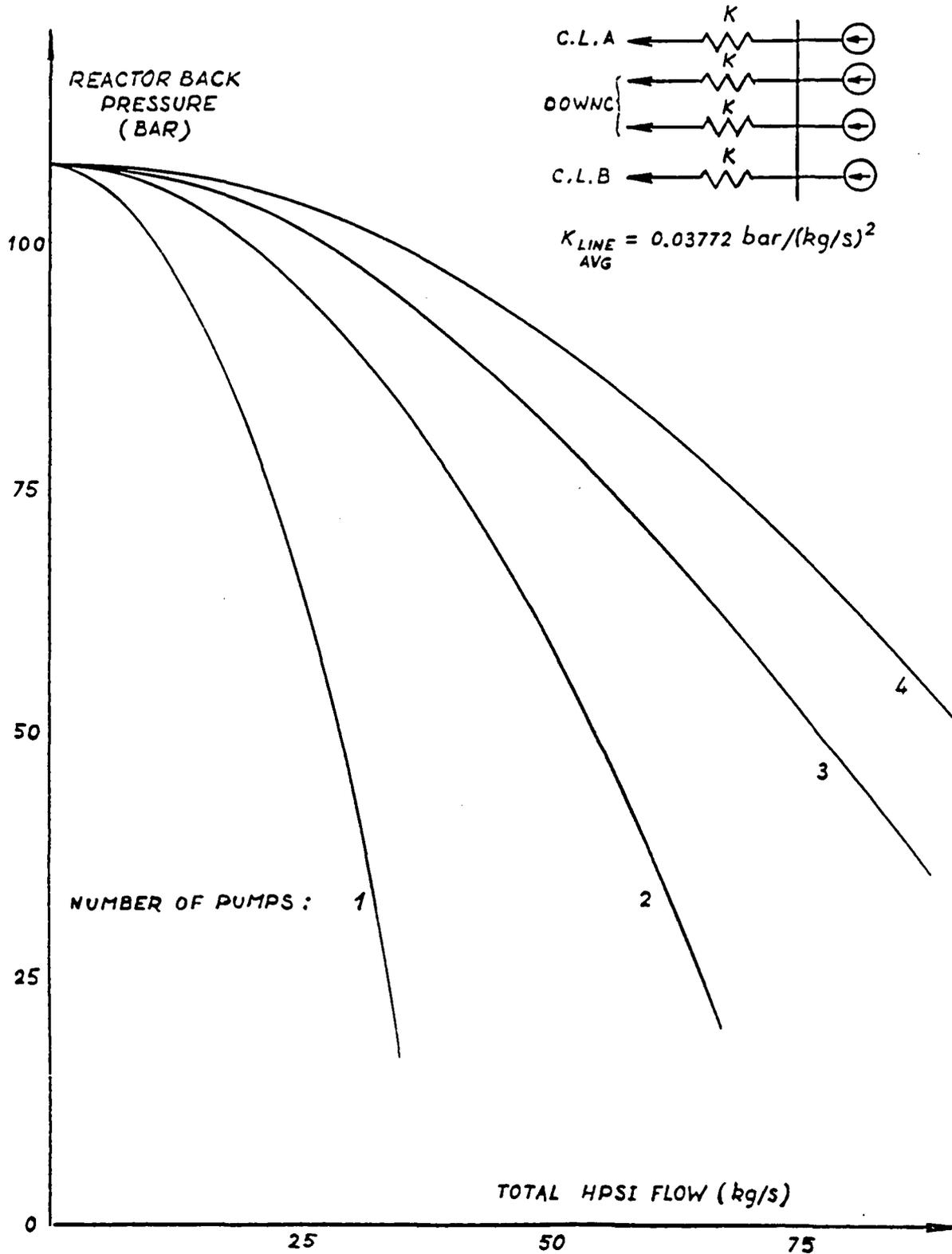


FIG 2.4 - DOEL 2 HPSI ECCS FLOW DELIVERY CURVES FOR TWO LINES TO BOTH COLD LEGS OPEN.

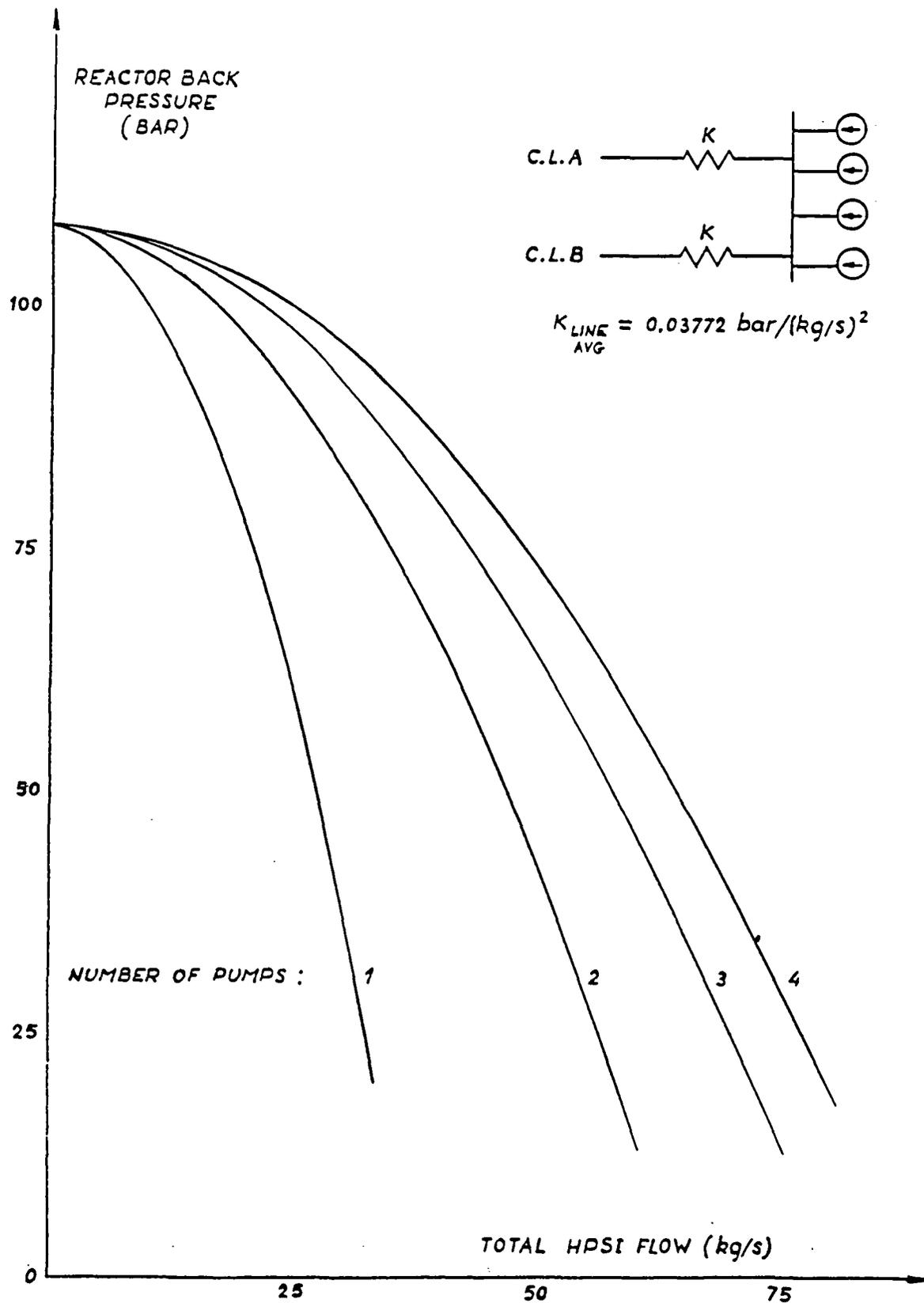


FIG 2.5 - HPSI PUMP CHARACTERISTIC (BEST ESTIMATE)

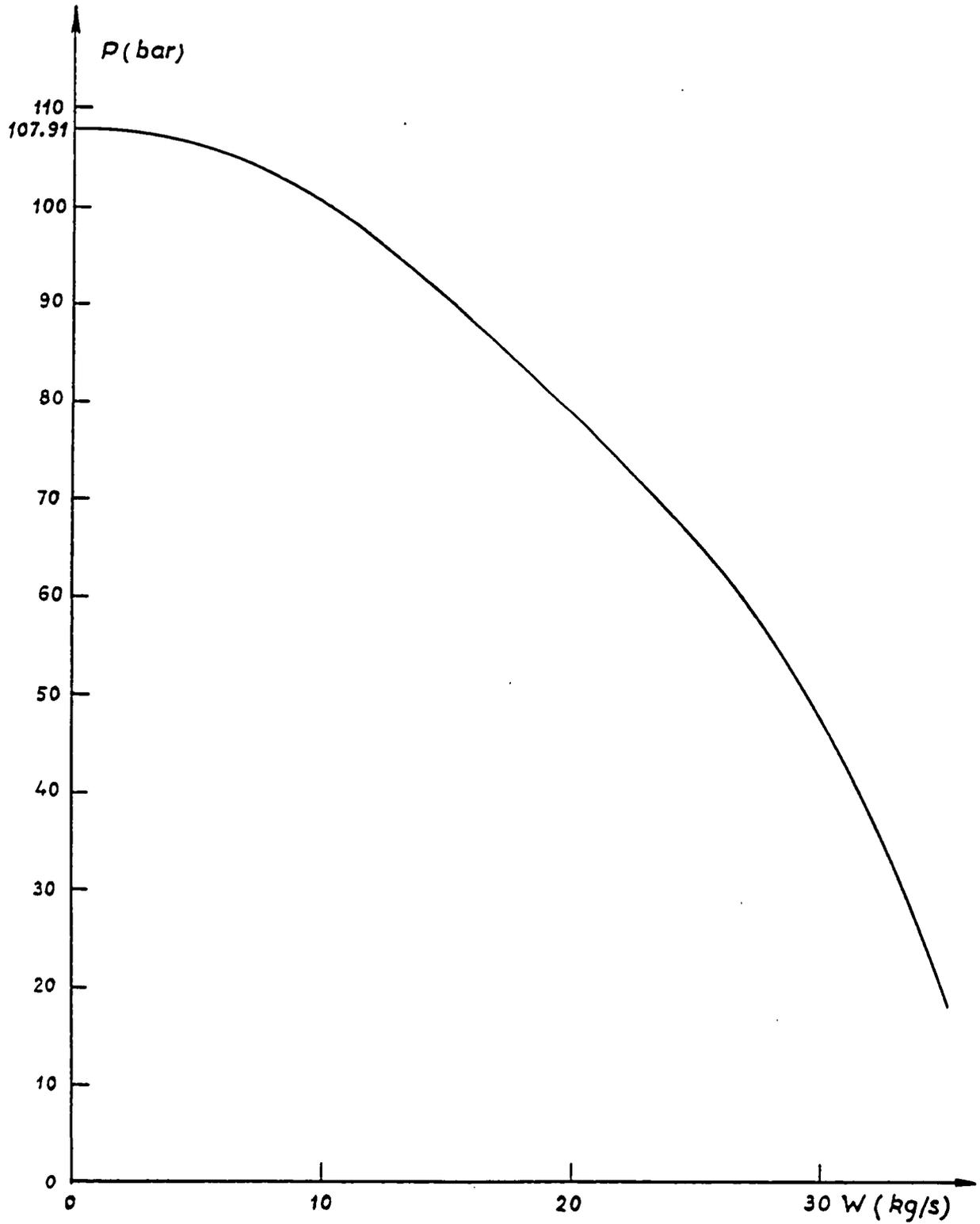


FIG 2.6: FLOW DIAGRAM OF THE AUXILIARY FEEDWATER SYSTEM

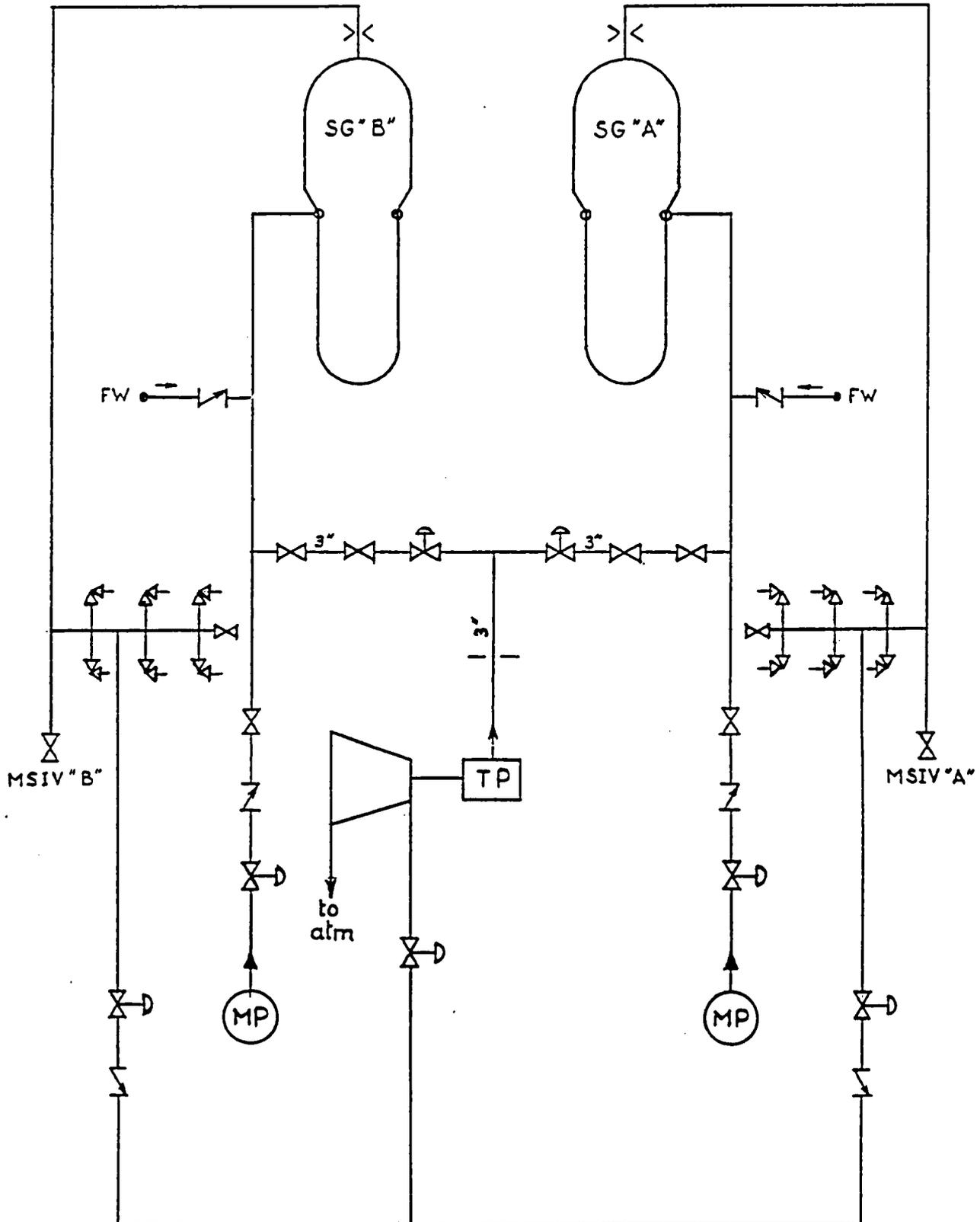
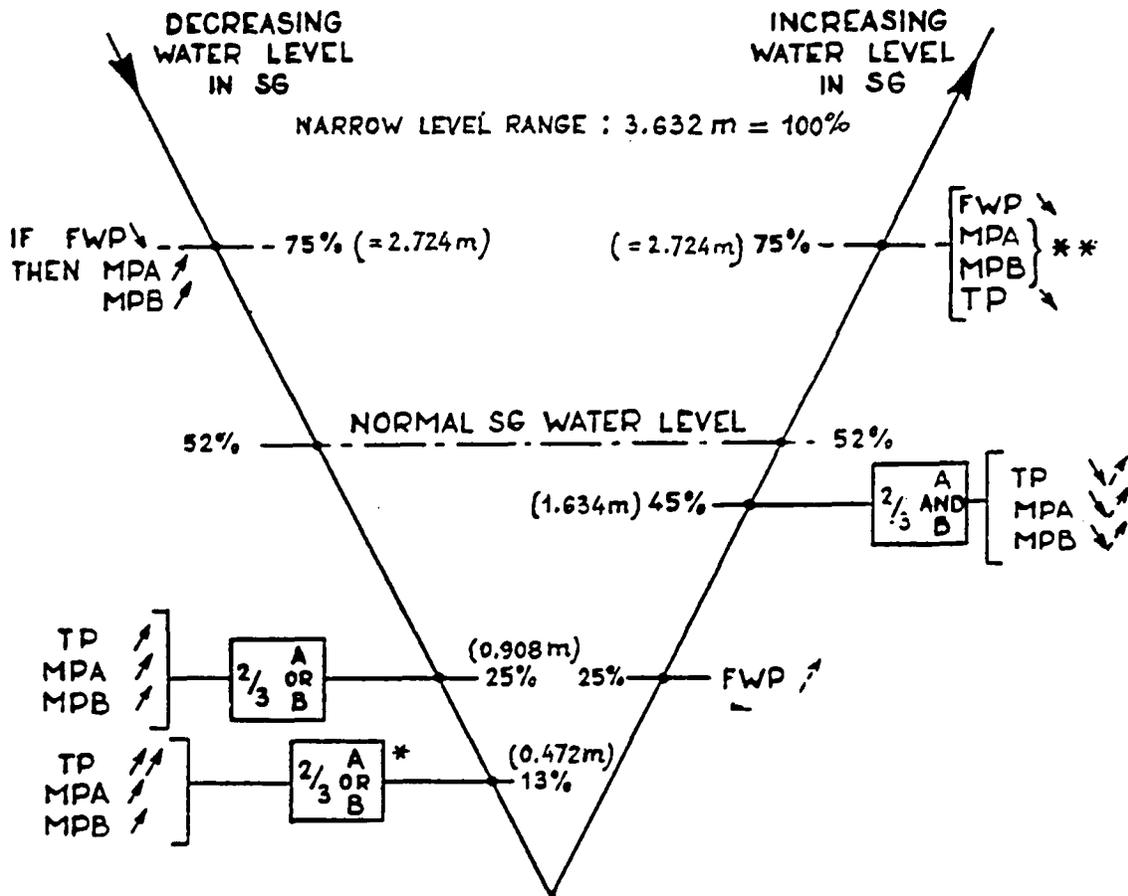


FIG 2.7 : AUXILIARY FEEDWATER ACTUATION TRIP

(for power levels > 20 %)

REF : DOEL 1708/95 REV. B (20.01.77). VOL. 1-CHAP.10



* : IF ALSO FW TEMPERATURE < 150°c
THEN FWP ↘, FW VALVES ↘ FOR BOTH SG.

** : IF "SI" OR "DIESELS" TRIP
IS SEQUENCED BY VITAL
BUS LOADING LOGIC

SYMBOLS : FWP : Feedwaterpumps
MPA : Motopump A
MPB : Motopump B
TP : Turbopump

↗ : Automatic start

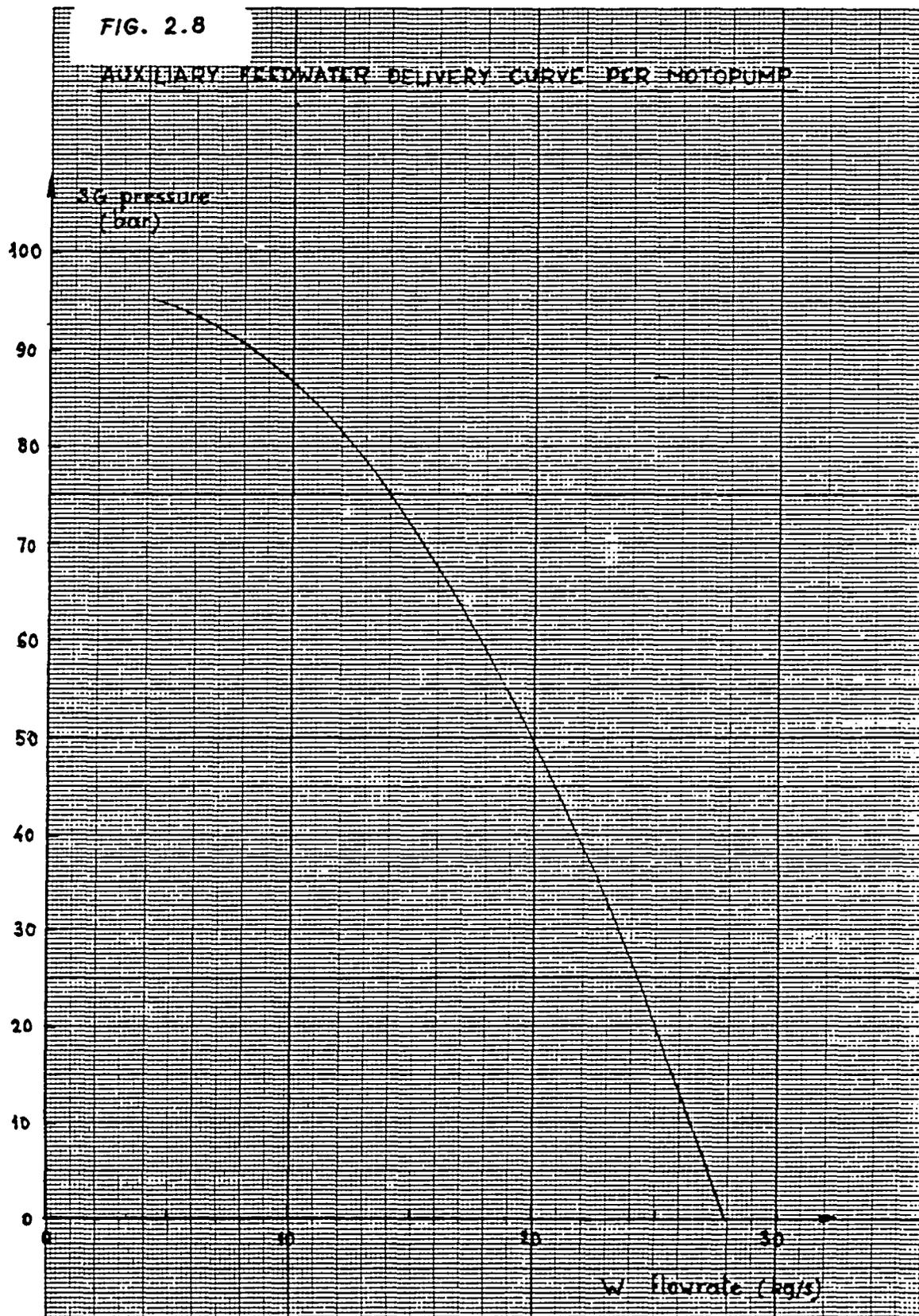
↘ : Automatic TRIP

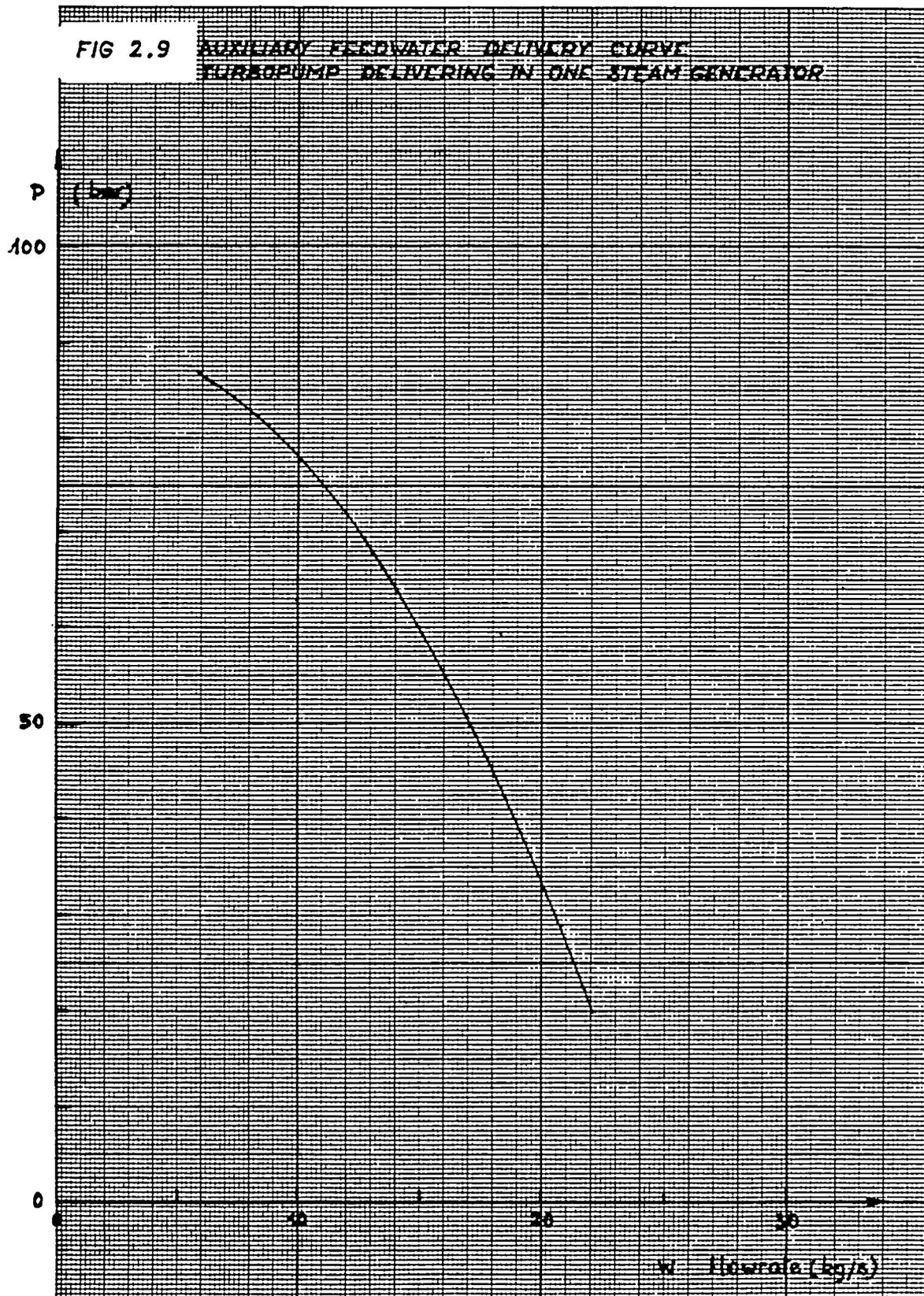
↗↗ : Can't stop manually

↘↘ : Can't start manually

↗↗ : Can start manually

↘↘ : Can TRIP manually





3. ANATOMY OF THE PLANT TRANSIENT

Part of the material presented in this chapter, and chapters 5, 6 and 7 has been presented before at two meetings in the U.S.A. (References 2 and 3). Information presented in these references and essential for code assessment is repeated in this document for completeness sake.

A report on the incident was issued on October 25th, 1979 (see annexe 1). From the raw plant recordings (figures 1 to 7 of the annexe), figure 3.1 was reconstructed for the most important plant variables.

From the original plant computer records, relevant computer diagnostics were retrieved and shown in table 3.1.

From this information, the best estimate sequence of events and operator interventions, as shown in fig. 3.2, was reconstructed, and is used as the basis for the event simulation.

3.1. Plant status prior to the SGTR (fig. 3.1 prior to point A)

At the moment the event occurred on June 25th 1979, the primary system was being heated up after a 24 hour stop for repair work on the main steam isolation valves.

The pressure had reached the rated value of 155 bar, with a RCS temperature of about 255°C.

The water level in the pressuriser was kept constant at 25 % by the automatic control of the charging flow. For the letdown, two orifices of about 10t/hr each were available.

The reactor was subcritical with all control rods down. Both primary pumps were running and the pressuriser heaters were ON. On the secondary side, the steamlines were both isolated by the main steam isolation valves and no condenser vacuum was

present. The main feedwater pumps were not operational and water level in both S.G.'s was manually controlled around 29 % level narrow band by means of the blowdown. The auxiliary feedwater pumps were not running.

The total heating power was about 11 MWth (2.5 MWth per primary coolant pump and 6 MWth decay heat).

3.2. Initiating event : (fig. 3.1, between points A and D)

At 19.20 hr (= t.) a quick level decrease in the pressuriser and a pressure decrease (≈ 2.5 bar/min) in the RCS was observed followed by a demand for increased charging capacity, and closure of the letdown line when the pressuriser level dropped below 20 %

While the pressuriser level recording went off scale (point B) a quick level increase in the B loop steam generator was observed (point C).

When the radiation monitoring channels of the SG blowdown recorded a maximum activity level, the operator diagnosed within a couple of minutes the cause of the event to be a leak in the B steam generator.

Post examination of the failed tube revealed a longitudinal crack of about 7 cm long located in the U bend and most probably caused by stress corrosion cracking. The initial break flowrate was about 15 kg/s (300 gpm).

3.3. Mitigation phase (fig. 3.1., between points D and L).

- . The faulted S.G. was completely isolated on the steam side, but the isolation of the steam discharge to the turbo-pump was omitted.
- . The atmospheric steam dump valve setpoint of the affected S.G. was set at maximum pressure to avoid steam release from this S.G.

- . A third charging pump was started in an attempt to compensate the leak rate.
- . By steam discharge to the atmosphere from the intact S.G. (point D), the operator started to cool down the plant, causing a decrease in temperature and a faster depressurization rate at t. + 15 min, in the RCS.
- . At t. + 18 min, the operator tripped the primary pump of the affected loop to reduce the RCS heat input.
- . At t. + 20 min, the safety injection signal was generated on low RCS pressure (117 bar) (point E) which caused the emergency diesels to start, initiated the containment isolation phase A, and caused ventilation isolation of the reactor building.
- . When reaching a pressure of 107 bar in the RCS (point F) all four HPSI pump injection started and stabilized the RCS pressure.
- . Steam discharge from the intact S.G. caused a water level drop below the low setpoint of 0.9 m on the narrow band level gauge (point G) and opened both steam discharge lines to the turbine driven auxiliary feedwater pump. This resulted in a quick pressure decrease of both S.G.'s (point H) followed by a rapid increase in the SG-A water level. The steam discharge from the affected S.G. through the turbine driven auxiliary feedwater pump was stopped about 8 min. later (POINT I) during which time about 1 ton of steam was released.
- . In an attempt to reduce the leak rate by equilibrating the RCS pressure to the affected SG, the operator started the primary pump B and used the full pressuriser spray capacity (point J) which caused a rapid drop of the RCS pressure. Vapour condensation in the pressuriser, combined with the full HPSI capacity caused a rapid increase in the water

inventory such that the pressuriser water level went off scale high (point K) at which time the operator stopped the pressuriser spray. This caused the pressure to increase from 75 bar to the cutt-off head of the HPSI pumps and stabilized at 107 bar (Point L).

3.4. Safety injection cancelling phase (fig. 3.1 between L and R)

A further primary pressure decrease was mandatory to

- . reduce the break mass flow rate and avoid flooding of the SG-B main steam lines;
- . avoid the opening of the safety valves of the faulted S.G;
- . start as soon as possible the shutdown cooling system (28 bar).

Therefore the operator first tried to cancel the S.I. signal in order to be able to trip the HPSI pumps. A circuit fault however did regenerate the S.I. signal on low RCS pressure after reset, each time requiring about 5 min before resetting. After about 20 min, the concerned bistables were flicked over manually which cancelled definitively the SI signal.

Three HPSI pumps were tripped (point M) and soon after (point N) the remaining pump was stopped after checking the subcooling margin. The RCS pressure dropped to about 65 bar (point O), for which pressure the charging system compensated the leak rate.

An attempt to open the letdown line failed as the isolation phase A also eliminated the compressed air supply in the reactor building.

It took the operator about 20 min to restore the air supply and to open the pneumatic isolation valves of the letdown line (POINT P).

After stopping a charging pump (point Q) the pressure decreased to the point where the residual heat removal system could be coupled to the reactor coolant system (point R).

TABLE 3.1.

SGTR : CHRONOLOGY OF EVENTS (PLANT COMPUTER)

Actual time	Elapsed time (s)	Computer diagnostic	Comment or set point
19.20.00.00	0.0	(rupture assumed)	$L_{PR} = 25 \%$
19.22.50.47	170.47	demand for higher charging flow	$L_{PR} < 25 \%$
19.23.15.67	195.67	pressure in pressuriser below reference	$P_{REF} = 155 \text{ bar}$
19.23.30.67	210.7	PR heaters OFF	$L_{PR} \ll 20 \%$
19.23.32.17	212.2	1st letdown orifice (closed/2CV024)	$L_{PR} \ll 20 \%$
19.23.32.77	212.8	2nd letdown orifice (closed/2CV025)	$L_{PR} < 20 \%$
19.23.36.37	216.4	Pressuriser low level alarm	$L_{PR} < 20 \%$
19.23.37.57	217.6	Letdown valve 2CV022 closed	$L_{PR} < 20 \%$
19.23.41.77	221.7	Pressuriser block valve 2PRO08 closed	$P \leq 145 \text{ bar}$
19.25.42.37	342.4	Spray valve PRO08 closing	$P \ll 145 \text{ bar}$
19.26.14.05	374.0	Low RCS pressure alarm	$p < 155 - 1.8 \text{ bar}$
19.30.29.77	629.8	Low-low pressuriser pressure alarm	$p \leq 147 \text{ bar}$
19.30.30.07	630.1	High level SGB (1/3)	$L_{SGB} \geq 65 \%$ (= 2,36m)
19.30.31.28	631.3	High level SGB (2/3) ($L \geq 65 \%$)	$L_{SGB} \geq 65 \%$
19.35.37.87	937.9	Computer saturation starts start 3rd charging pump start cooldown with SG	} assumed
19.37.19.57	1039.6	Computer saturation stops	
19.38.31.65	1111.7	Reactor coolant pump B tripped	normal
19.38.59	1139.0	Low pressure in SGA	
19.40.17.88	1217.9	Pressuriser pressure $< 118.5 \text{ kg/cm}^2$	$P_{PR} = 117 \text{ bar}$
19.40.19.18	1219.2	SCRAM (from SI generation)	
19.40.19.18	1219.2	Diesels start	
19.40.19.27	1219.3	High flow in MSB (at venturi)	

TABLE 3.1.
(continued)

SGTR : CHRONOLOGY OF EVENTS (PLANT COMPUTER)

Actual time	Elapsed time (s)	Computer diagnostic	Comment or set point
19.40.19.27	1219.8	Ventilation isolation in containment	SI
19.40.19.57	1219.6	Containment isolation (phase A)	SI
19.40.23.49	1223.5	Start command SI pump D	SI
19.40.23.61	1223.6	SI pump D started	
19.40.23.66	1223.7	Start command SI pump A	SI
19.40.23.75	1223.8	SI pump A started	
19.40.23.77	1223.8	Isolation of compressed air	SI
19.40.24.19	1224.2	Safety injection alarm	SI
19.40.24.20	1224.2	Start command SI pump C	SI
19.40.24.31	1224.3	SI pump C started	
19.40.24.45	1224.5	High MS flow B stopped	
19.40.24.49	1224.5	Start command SI pump B	SI
19.40.24.60	1224.6	SI pump B started	
19.40.33.07	1233.1	Valve 2SI011 open	Cold leg A
19.40.33.37	1233.4	Valve 2SI009 open	Cold leg B
19.40.33.67	1233.7	Valve 2SI127 open	Downcomer
19.40.34.87	1234.9	Valve 2SI125 closed	Downcomer
19.42.04.27	1324.4	Computer saturation starts	
19.42.55.87	1375.9	Computer saturation stops	
19.43.48.32	1428.3	Very high AFW flow (2FW-51) to SGA	
19.44.02.47	1442.5	Low level in SGA	L \ll 25 % (0,9 m)
19.44.39.14	1479	Very high AFW flow to SGB	
19.44.42.37	1482	Steam admission valve (SGB) to turbopump open	TP starts
19.44.50.77	1490.8	Steam admission valve (SGA) to turbopump open	TP starts
19.46.49.24	1609.2	Very high AFW to SGB	

TABLE 3.1.

(continued)

SGTR : CHRONOLOGY OF EVENTS (PLANT COMPUTER)

Actual time	Elapsed time (s)	Computer diagnostic	Comment or set point
19.46.22.70	1582.7	Low pressure in SGB	
19.46.28.87	1588.9	Low level in SGA	L _{SGA} < 25 %
19.46.40.05	1600.0	Low pressure in SGB	
19.48.01.15	1681.1	High AFW to SGB stopped	Trip turbopump
19.48.02.00	1682.0	Computer saturation starts	
19.53.12.00	1992	Computer saturation stops	
19.53.12.44	1992.4	Motopump B disarmed	Trip motopump B
19.55.05.72	2105.7	Start of primary coolant pump B	manual
19.56.33.67	2193.7	Reduce charging pump speed	P _R level normal
19.57.10.87	2231	Pressuriser level normal	L _{PR} > 25 %
19.57.28.57	2249	PR heaters on	
19.58.09.67	2287	Spray valve 2PR001 closed	} spray stopped
19.58.12.97	2293	Spray valve 2PR002 closed	
19.58.33.07	2313	Spray valve 2PR001 not closed	} spray on
19.58.33.67	2314	Spray valve 2PR002 not closed	
19.58.47.77	2328	High pressuriser level	L _{PR} ≥ 30 %
19.59.37.17	2377	Very high pressuriser level	L _{PR} ≥ 70 %
20.00.15.15	2415	Reset command all SI pumps	
20.00.15.16	2415	SI cancelled	
20.00.21.05	2421	SI generated	
20.03.20.77	2601	Spray valve 2PR001 closed	
20.03.24.07	2604	Spray valve 2PR002 closed	
20.03.52.57	2633	High level SGA	L _{SGA} ≥ 65 %
20.05.05.00	2705	High level SGA (2/3)	L _{SGA} > 65 %

FIG 3.1

EVOLUTION OF SOME IMPORTANT SYSTEM PARAMETERS DURING TRANSIENT

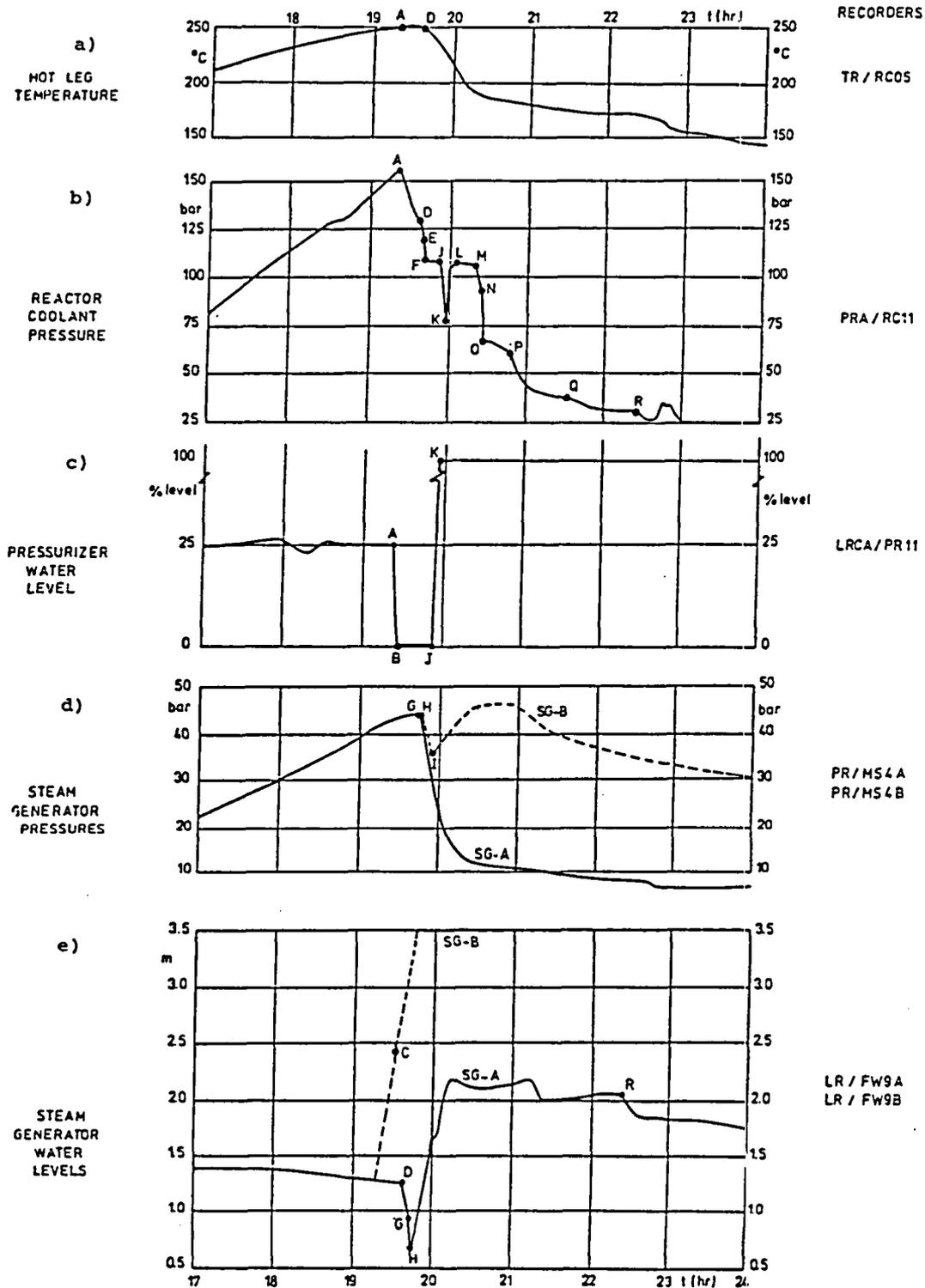
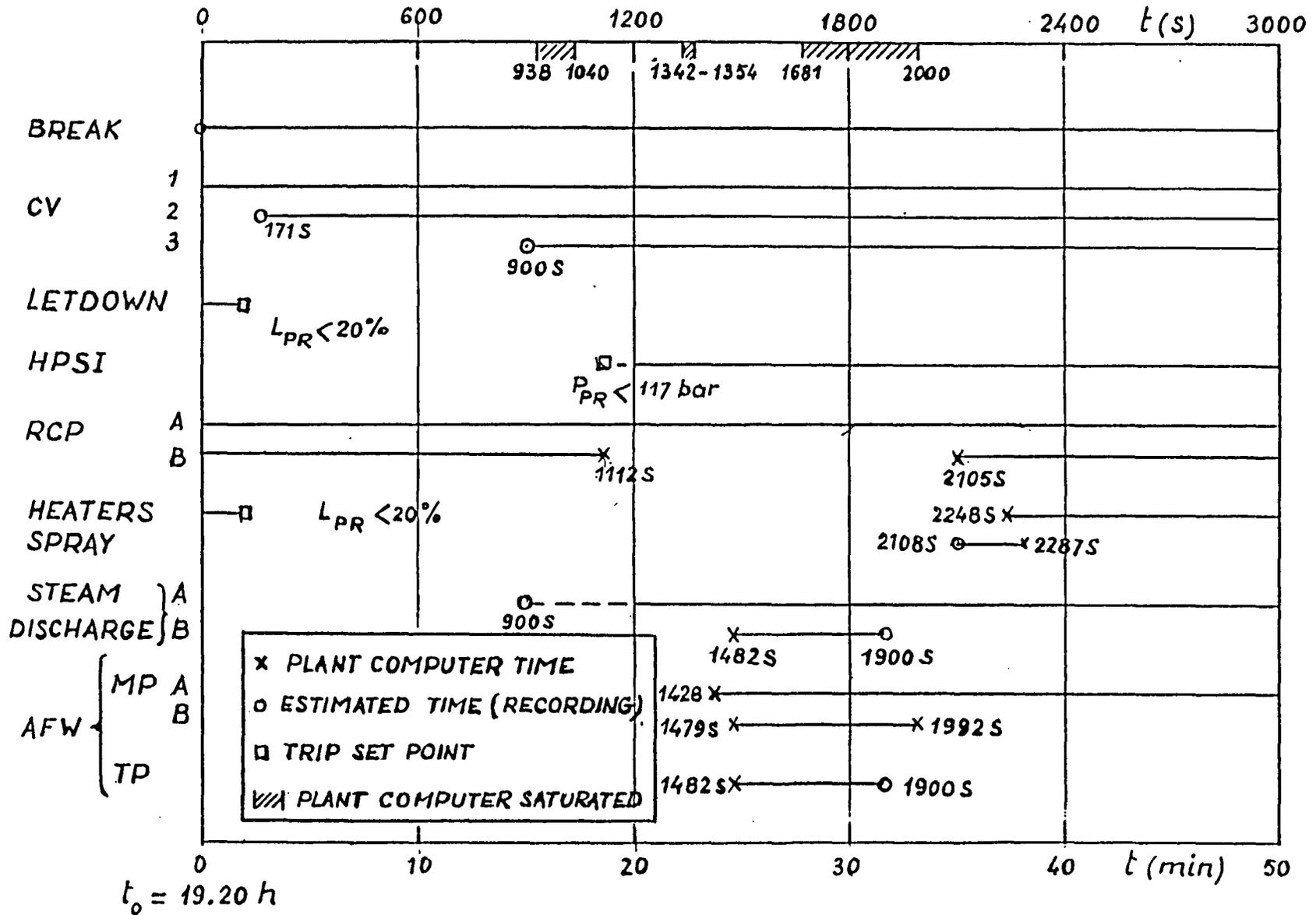


FIG 3.2 SEQUENCE OF EVENTS (DOEL 2 - SGTR 4)



4. CODE AND MODEL DESCRIPTION FOR PLANT SIMULATION

The code used was RELAP-5 MOD-2 cycle 36.01 and was processed on a CDC machine CYBER 176.

The simulation was performed over a period of 2700 s, starting at the estimated time of rupture at $t_r = 19\text{hr } 20\text{ min}$. (Point A on fig. 3.1). This period covers the most interesting plant transient phenomena.

The input data deck for the base case is joined in annexe 2.

In order to understand and interpret the main parameters of the input data deck, the general and special nodalisation features are described in this chapter.

The general nodalisation philosophy is a compromise between precision and economy. For the primary system, a minimum transport time of 0.15 s was selected, at nominal conditions, leading to a four volume representation of the core. The minimum length of the loops was taken to correspond to the length of the horizontal section of the cross over legs. In the U tube bundle, the number of axial nodes is based on a rule of thumb of about $2^\circ \Delta T$ per volume, in order to obtain an acceptable heat transfer and temperature drop between primary and secondary. This resulted in 10 axial nodes in the upgoing leg, and 5 axial nodes in the descending leg of the U tube bundle. This also sets the volume length in the riser of the S. G.

The simulation of the S. G. separator and dryer region was principally based on the RELAP-5 requirements for the mechanistic separator model, as recommended in the manual (Ref. 4).

The pressuriser nodalisation was chosen fine enough to accommodate large insurges and outsurges. All important structures were modeled as heat slabs in order to properly account for the important structural heat exchange between

structures and fluid during important temperature (or pressure) changes in the system.

For completeness, also the heat release to the containment was accounted for.

This resulted in the nodalisation scheme as shown in fig. 4.1., consisting of 154 volumes, 158 junctions and 164 heat slabs (563 mesh points).

The remainder of this chapter explains the detailed nodalisation features for the

- . reactor vessel
- . primary loops
- . S.G.
- . pressuriser.

Furthermore, special nodalisation features are described for

- . the leak simulation
- . the high pressure safety injection system
- . the charging and letdown system
- . the pressuriser spray and heaters
- . the S.G. atmospheric steam dump valve
- . the auxiliary feedwater system.

4.1. Nodalisation of the primary coolant system and S.G.

The numbering logic for the RCS system is as follows :

- | | |
|---|-----------------|
| a. reactor vessel components | 010 - 099 |
| b. B loop components | 100 - 199 |
| c. A loop components | 200 - 299 |
| d. pressuriser components | 300 - 399 |
| e. Steam generator (secondary components) : | 600 - 699 (SGB) |
| | 700 - 799 (SGA) |

Table 4.1. compares the calculated steady state conditions at 100 % power with the nominal plant values for most relevant parameters.

Table 4.2. summarises the geometrical data, in terms of free volume, coolant mass inventory at 100 % power and the structural mass inventory which are compared to nominal plant data (where available).

4.1.1. Reactor vessel nodalisation (volumes 011 - 099)

Table 4.3. summarises the dimensions of the hydrodynamic components corresponding to the volume arrangement shown in fig. 4.2.

Table 4.4. summarises the individual pressure losses in the reactor vessel from which the junction form loss factors are derived.

4.1.2. Nodalisation of the primary loops (volume numbers 111 - 153, 211 - 253)

Tables 4.5 and 4.6 summarize respectively the dimensions of the hydrodynamic components for the loops B and A.

The nominal pressure losses in primary loops (hot leg, intermediate legs and cold legs) amount to 0.31 bar.

Additional form loss factors have been added in the junctions of the intermediate legs, such that the frictional plus form losses in the primary piping add up to this nominal value.

4.1.3. Nodalisation of S.G. U tube bundle (volumes 121 - 125, 221 - 225).

The detailed nodalisation for 1 SG is shown in fig. 4.3. and table 4.7 summarizes the dimensions of the hydrodynamic components on primary and secondary side.

As seen in table 4.7, all U tubes have been globalised into one pipe with 10 volumes on the hot side (122/01 - 122/10) and 5 volumes on the cold side (124/01 - 124/05) with a total area of about 1 m². For the simulation of the leak, the volumes 122/09 and 122/10 have been replaced by 1 branch 123 in order to accommodate the leak junction.

The frictional pressure losses in the U tube bundle are based on a drawn tubing roughness of 0.1 micron and a hydraulic diameter 19.66 mm (inside diameter of 1 tube).

Additional form loss factors for contraction and expansion are added for the hemispherical bottom of the SG primary sides, in order to obtain a total pressure loss of 1.56 bar between inlet and outlet of a steam generator.

4.1.4. Simulation of the primary pumps (Vol. 140, 240).

The pump geometrical data are summarised in table 4.5.

Inlet-outlet form losses :

The pump inlet and outlet form losses are chosen to obtain the nominal loop pressure drop corresponding to the nominal pump head.

ΔP_f pressure vessel	: 2.16 bar
ΔP_f steam generator	: 1.56 bar
ΔP_f primary loops	: 0.31 bar
ΔP_f pump inlet-outlet	: 0.295 bar

	4.325 bar
Buoyancy head	- 0.050 bar

Nominal net pump head	: 4.275 bar

Pump hydraulic data :

- . Since not enough data are available, the built-in WESTINGHOUSE PUMP model was selected as a best guess.
- . The pump hydraulic torque, the motor torque and the frictional torque at nominal speed (990 RPM) are based on the nominal pump head (58 m), the volumetric flowrate (4.826 m³/s) and the pump gross power (2.44 MW).
- . The frictional torque at zero was taken from the measured value for cold water and adjusted to hot water conditions.

4.1.5. Pressuriser nodalisation

Table 4.8. summarises the geometrical data for the hydrodynamic components of the pressuriser and the surgeline.

The pressuriser spray is taken from volumes 152 and 252 in the cold legs and forced in volume 317 of the pressurizer, by means of 2 time dependent junctions (J345, J346) to correspond to a nominal maximum spray flow rate of 1500 l/min when spray valves are fully opened.

Fig. 4.4 provides pressuriser level data.

The pressuriser level taps are connected to a ΔP cell which effectively compares the weight of a reference water column to the weight of the water (and steam) in the pressuriser.

The electrical signal, corresponding to a pressure difference varies between 0 and 100 % for water level varying between lower tap and upper tap. Hence, one can write.

$$\text{Level (\%)} = 100 \times \frac{\Delta P_z - \Delta P_o}{\Delta P_{100} - \Delta P_o} = 100 \times \frac{\sum_i \rho_i g h_i - \rho_v g H}{\rho_e g H - \rho_v g H}$$

wherein : g = earth gravity

ρ = density

H = distance between level taps (= 8.243 m)

h = height of hydrodynamic volume

subscript i = index of hydrodynamic volumes in pressuriser

z = water level

v = vapour at saturated conditions

l = liquid at saturated conditions

0 = 0 %

100 = 100 %

The pressuriser is equipped with various level measurement cells which are of two types :

a. cold calibrated at atmospheric pressure and 60°C.

In this case the values of the densities of water and vapour are constant at 1 bar and 60°C.

$$\text{Hence } L_c (\%) = 100 \times \frac{\sum_i \rho_i g h_i - \rho_v^c g H}{\rho_l^c g H - \rho_v^c g H} \quad (a)$$

superscript c : cold calibrated conditions

b. Hot calibrated conditions at nominal power :

p = 155 bar (saturation)

$$\text{Hence } L_h (\%) = 100 \times \frac{\sum_i \rho_i g h_i - \rho_v^h g H}{\rho_l^h g H - \rho_v^h g H} \quad (b)$$

superscript h = hot calibrated conditions

Since there is no adjustment for different fluid temperatures in the pressuriser, there can be a strong difference between the real (collapsed) water level and the measured level with the hot or the cold calibrated cells. This relationship is

shown in fig. 4.5 for different fluid conditions in the pressuriser.

By means of the control variables, the various water levels are calculated in the program :

CNTRLVAR 900 : calculates the collapsed water level

CNTRLVAR 901 : calculates L_a (expression (b) above)

CNTRLVAR 902 : calculates L_c (expression (a) above)

using the summation control variable with

	HOT	COLD
gain = $\frac{1}{\rho_{gH} - \rho_{v gH}}$	$2.515 \cdot 10^{-3}$	$1.2459 \cdot 10^{-3}$
constant = $-\rho_{v gH}$	- 8248 Pa	0

4.1.6. Nodalisation of the S.G secondary side.

Fig. 4.3. illustrates the correspondence between real S.G. volumes and RELAP-5 nodalisation.

Table 4.7. summarises the geometrical data for the various hydrodynamic components of the S.G. and the main steamlines up to the MSIV valves.

The single junction forward form loss factors are chosen in order to obtain a nominal recirculation of 6. This is mainly controlled by the junctions 617 ($K_f = 3.3$), 62301 ($K_f = 20.4$), 62302 ($K_f = 1.0$), and 62303 ($K_f = 1.81$) around the separator.

Since strong reversed natural circulation was observed during some calculation, very large reverse form loss ($K_f = 1000$) factors were introduced in the downcomer, the junctions into the feedwater inlet plenum and the separator.

The S.G. outlet venturi was modelled in the outlet junction 630/01 with effective venturi area of 0.13 m² ($K_F = 0.2269$, $K_R = 0.4969$).

4.2. Special code models and options

This paragraph deals with some code models which were incorporated especially for simulating the .SGTR event.

4.2.1. Steam generator B tube leak model.

The break junction is modelled as a single junction (J618) connecting the outlet of a pipe component (V 122) on the primary side to a branch component (V 621) on the secondary side. The cross flow junction option ($S = 3$) was used with an orifice type form loss ($K_F = K_R = 2.6$). With these options, the area of the junction was adjusted to a final value of 1.724 cm² for which the calculated water level rise in the affected S.G. matched the recorded data (fig. 3.1 - e). Since the plant was close to a hot shutdown condition without any normal feedwater addition, the resulting water level signal demonstrated almost a straight line from which the leak rate could be determined a posteriori with acceptable precision. The initial break flowrate determined in this way amounts to 14.9 kg/s (= 300 gpm), and the adjusted area of 1.724 cm² corresponds to about 57 % of the crosssectional area of 1 S.G. tube.

4.2.2. High pressure safety injection

From the plant computer information, it turned out that 1 SI valve for downcomer injection closed at $t = 19.40.34.87$. Shortly afterwards the plant computer was saturated during 12 seconds and it is not known if the valve was opened manually. However, since the RCS pressure during the SI period stays close to the cut-off pressure of the HPSI pumps, one should infer that downcomer injection did take place with 4 SI lines to the primary.

Hence the programmed HPSI delivery curve used corresponds most likely to the configuration of figure 2.3. Owing to the uncertainty of direct downcomer injection, only two SI injection points were modelled in both cold legs. Hence, coldwater taken from TDV (170, 270) at 15°C, is injected for 50 % in cold leg A (TDJ 271) and 50 % in cold leg B (TDJ 171) with a RCS pressure flowrate dependance corresponding to the flow delivery curve of fig. 2.3 (4 pumps). The table trip number 571 simulates the generation of a SI signal when pressure in pressuriser drops below 117bar. The SI flowrate look-up table consists of a compensated RCS pressure (CNTRLVAR 070) and SI flowrate.

Instead of taking the local pressure at the injection points, for controlling the SI flow, the upper plenum pressure was taken (V043) combined with a numerical smoothing as follows :

$$P_{(SI)} = 0.9 P_{OLD} + 0.1 P_{043(NEW)}$$

This technique may introduce a lag time (7.5 seconds to 96 %) but avoids strong fluctuations in the SI flow due to strong pressure feedback between injection volume pressure and corresponding SI flow.

4.2.3. Steam generator A atmospheric steam dump valve

The steam relief capacity of this valve corresponds to 5 % of the total plant steam flowrate of 666 kg/s. The valve (V791) was modelled as a motorvalve for which the flow area was adjusted to yield the nominal relief capacity at nominal pressure (i.e. 33.3 kg/s at 72 bar) with $K_F = K_R = 1.0$.

The valve was programmed to open with a constant rate between 900 and 1200 s. The exact actuation of this valve is not recorded. In order to reduce excessive water level swell by sudden opening of this valve, a duration of 300 s was selected (see also chapter 7).

4.2.4. RCS charging and letdown system.

The charging and letdown flowrate were not monitored during the event, hence a best estimate reconstruction has been implemented, based on the plant conditions at the time of the break, and the plant computer data.

The plant is equipped with three volumetric charging pumps. Per pump, the minimum charging flow is $8.7 \text{ m}^3/\text{hr}$ ($= 2.42 \text{ kg/s}$) at 140 RPM
 the maximum charging flow is $14.7 \text{ m}^3/\text{hr}$ ($= 4.08 \text{ kg/s}$) at 234 RPM

One assumes that at the moment of the tube rupture, one charging pump was operational at 50 % flow capacity or 3.2 kg/s.

At an elapsed time of 170 s, the plant computer signals a need to start an additional charging pump, from which is inferred that the maximum charging capacity of 4.08 kg/s is reached in 170 s, and that a second charging pump is actuated at full speed (additional 4.08 kg/s). A third charging pump is added around 900 s (estimated time, since computer was saturated between 938 s and 1 040 s), for which again full speed was assumed in order to cope with the primary coolant inventory loss.

From the computer information, it is known that 2 letdown orifices of each 10 t/hr in nominal conditions, were open at time of the tube rupture. Furthermore, there exists also a leak path via the primary pump seals back to the CV tank.

The initial letdown flow was estimated from the required charging - letdown flow mismatch during the heat-up phase in order to maintain a constant water level in the pressuriser.

A total thermal heat input of 11.85 MWth (decay heat + 2 primary coolant pumps + pressuriser heaters) at a RCS pressure of 155 bar and 255°, would lead to an isobaric volumetric expansion of $4.38 \cdot 10^{-3} \text{ m}^3/\text{s}$, which can be compensated by an excess letdown of 3.6 kg/s (density = 804 kg/m^3). Hence the

initial letdown flow was taken at $3.6 + 3.2 = 6.8$ kg/s and maintained constant until the letdown closes at 170 s.

It should be recognised that a maximum net charging flowrate of max. 12.24 kg/s is not much smaller than the initial estimated leak flow of 14.9 kg/s. The resulting pressuriser level evolution could hence be affected by a very large uncertainty.

4.2.5. Pressuriser spray and heaters

The pressuriser heaters are incorporated as a heat source of 850 kW in the heat slab bounding pressuriser volume 312. The actuation logic for the heaters is such that the heaters are switched off when the water level drops below 20 % (variable trip 580) and are switched on at an elapsed time of 2248 s (variable trip 542) conform to the plant computer data (fig. 3.2).

The pressuriser spray lines are simulated explicitly, except for the spray flowrate which is controlled, not by the pressure head of the primary coolant pumps, but by 2 time dependent junctions (J 345, 346) in order to force a nominal spray flow of 1500 l/min when the spray valve is opened (see nodalisation on fig. 4.1.). The spray flow is injected in volume 317 which corresponds roughly to the elevation of the pressuriser spray nozzle.

From the computer data, there is no trace for the opening time of the spray. However, since the operator started the second primary coolant pump (at an elapsed time of 2106 s) in order to use full spray, it is assumed that the spray was initiated a few seconds after starting the primary pump (variable trip 529) and stopped at an elapsed time of 2287 s (variable trip 530) as registered by the plant computer.

4.2.6. Auxiliary feedwater system

The auxiliary feedwater motordriven pumps (one for each SG) are modelled by a TDV (605, 705) and a TDJ (607, 707) which inject cold water (35°C) in the SG's, at a rate controlled by the SG pressure (volumes 630, 730) conform to the delivery curves for each motordriven pump (fig. 2.7).

For the turbine driven pump, the auxiliary feedwater flow to both SG's and the steam flow to the turbine is evaluated by means of a complex turbine driven auxiliary feedwater pump control block (Fig. 4.6) conform to the flow schematic of fig. 2.6.

The turbine driven auxiliary feedwater pump flow to each SG depends on the SG back pressure (P_A , P_B) the pump head (P_p) and the flow resistances in the common branch, ($K_{p,c}$) and both individual branches ($K_{p,a}$ and $K_{p,b}$) as follows :

$$W^2_{p,a} = \frac{P_p - P_A - K_{p,c} W^2_p}{K_{p,a}} \quad (a)$$

$$W^2_{p,b} = \frac{P_p - P_B - K_{p,c} W^2_p}{K_{p,b}} \quad (b)$$

Assuming a known turbine driven auxiliary feedwater pump head (P_p) and a total pump flow (W_p) from former cycle, the individual flows can be evaluated by means of control blocks 865,866 and 868 conform to eq. (a), and by means of control blocks 865,867 and 869 conform to eq (b). The total turbine driven auxiliary feedwater pump flow is then obtained in the adder 872.

Knowing the total pump flow, one then can evaluate the pump head (P_p) from the known pump characteristic :

$$P_p = \left(\frac{n}{n_o}\right)^2 P_{p,o} - K_p \left[W_p - \left(\frac{n}{n_o}\right) W_{p,o} \right]^2 \quad (c)$$

wherein $P_{p,o}$ is the maximum pump head

$W_{p,o}$ is the net pump flowrate at max. pump head

K_p is the pump loss factor

n/n_0 is the speed ratio of the turbine driven auxiliary feedwater pump

The expression (c) is evaluated in control blocks 880, 881 and 882. Knowing the total pump flow (W_p in block 872), one can also calculate the required pump power in blocks 874 and 875 conform to the expression (d) :

$$V_p = \left(\frac{n}{n_0}\right)^3 \left[V_{p,f} + d \left(\frac{n_0}{n}\right) W_p \right] \quad (d)$$

Wherein $V_{p,f}$ is the power required at zero flow (friction)
 d is the power coefficient for given pump flow.

At this point one has to verify if the required pump power does not exceed the maximum turbine power which is controlled by the available steam flow to the turbine. Since such turbine is supposed to yield full power for a large range of steam pressures, the available steam from the SG is throttled down to 9 bar at the turbine inlet valve and one can reasonably assume that down to 9 bar in the SG's, the turbine can provide full power, which is rated at 368 kW.

Based on the general Rateau laws, one assumes that the power varies with the third power of the speed, and for required pump power above 368 kW (from d), the new speed ratio becomes:

$$\frac{368}{V_p} = \left(\frac{n}{n_0}\right)^3 \quad \text{with limit : } n \leq n_0$$

The actual speed ratio is evaluated in control blocks 876, 877 and 878.

Finally, knowing the turbine power V_p , one can calculate the gross steamflowrate extracted from the SG at the highest pressure; according to the turbine power-flowrate characteristic (e) :

$$W_{v,t} = W_{v,s} + \frac{V_p}{\Delta H} \quad (e)$$

Wherein : $W_{v,t}$ = Vapour flowrate to turbine

$W_{v,o}$ = Vapour flow not producing work (dissipation)
 ΔH = enthalpy drop of steam expanding from 9 bar
to atmosphere.

This is evaluated in control block 883 which gives the steamflow extracted from 1 SG (at highest pressure).

Direct feedback from outlet to inlet is applied for stabilisation purposes of some vital parameters such as W_p (872), P_p (882) and V_p (875), whereby the new value at the actual time step is based for 50 % on the output value of the preceeding step and for 50% on the new input parameters.

The actuation control logic for the auxiliary feedwater motor driven pumps and turbine driven auxiliary feedwater pump was not based on the water level program of fig. 2.6, for 2 reasons :

First, the operator has taken over the auxiliary feedwater system in the manual mode for some periods in the transients, and secondly, to avoid the feedback of discrepancies in the calculation of the water level into the operation of the auxiliary feedwater system (see also chapter 7).

Hence, the actuation logic for this system was based on the recorded timing for the start of the auxiliary feedwater system, and on a best estimate timing for the trip of the turbine driven auxiliary feedwater pump (logical trip 661 which goes into trip units 862 and 863 of the turbine driven auxiliary feedwater pump control logic).

Table 4.1 : Comparison between system nominal parameters and calculated steady state conditions (after stabilization)

Component	Variable	Units	Nominal	Calculated	Comments
G013/02 , 03	Primary coolant flow	kg/s	3628.	3626.7	$p = 751.72$ fixed input
C140, C240	Pump head	m	58.	58.	
C140, C240	Pump speed	rad/s	103.67	103.67	
C153, C111	ΔP reactor vessel	bar	2.16	2.156	
C025, C041	ΔP core	bar	1.	0.992	
C113/01, C131/01	ΔP steam generator	bar	1.56	1.57	
(G011/02)/(G01302+03)	Vessel head bypass flow	%	0.24	0.25	
(G025/02)/(G023/01)	Core bypass flow	%	4.5	4.52	
T013	Vessel inlet temperature	$^{\circ}C$	287.3	287.3	
T043	Vessel outlet temperature	$^{\circ}C$	317.1	316.8	
T043 - T013	Δt core	$^{\circ}C$	29.8	29.6	
P112	System nominal pressure	bar	155.1	155.1	
CV610 + CV710	Total heat output	MW	1192.	1192.	

Table 4.1: Comparison between system nominal parameters and calculated steady state conditions (after stabilization)

Component	Variable	Units	Nominal		Calculated	Comments
J609, J709	Feedwater flowrate	kg/s	333.		333.	fixed input
J63401, 73401	Steam flowrate	kg/s	333.		332.6	
T 610	Feedwater temperature	°C	230.		230.	fixed input
D 630	SG steam pressure	bar	58.8		59.	
CV 611, 711	SG downcomer water level	m	11.662		11.69	
	recirculation ratio		(6)		5.7	
CV600,700	SG water mass	kg	36220		36030	

TABLE 4.2

		FREE VOLUME (m ³)			WATER MASS (t) AT 100% POWER			STRUCTURAL MASS (t)		
		RELAP	NOMINAL	%	RELAP	NOMINAL	%	RELAP	NOMINAL	%
1	Reactor vessel	56.302	56.77 56.	-0.8 +6.	40,453			300,64	214+90	1
2	Hot legs	2.845	2.825	+0.7	1,956			8.70		
3	Cold legs	3.015	2.933	+2.8	2,268			9.21		
4	Pump suction legs	3.251	3.25	0	2,443			9.86		
5	Pump casing	7.757	7.758 11.072	0 -29.9	5,330			51.99	52	-0.02
6	SG primary	26.762	26.74	+0.1	19,452			33.56		
7	Pressuriser	24.156	24.06	+0.4	2,480 1,007*			44.34	48	-7.6
8	Surge line	0.650	0.652	-0.2	439			2.87		
9	Total for primary coolant syst. (1 + 2 + 3 + 4 + 5 + 6 + 7 + 8)	168.368	170. 168.49	-1. +0.1	114,277			574.49		
10	SG secondary side	136.495	136.5	0	33.200 2.825*	1,176* 1,400 39,198	-0.54 -8.33	241.08		
11	Steam lines	97.132			2,900			99.91 103.37		
12	Collector	3.324			0.113			5.94		
13	Total for secondary side (10 + 11 + 12)	237.451			39,038			691.38		
14	Total for S.G component (6 + 10)	190.019	189.98	+0.02	55,477	58.	-4.35	279.64	280	-0.13
15	Grand total	595.138			209,792			1835.15		

TABLE 4.3 : Summary of reactor vessel nodalisation (volumes)
(STEADY STATE 11/85)

Component	Area	Length	Volume	Watermass	Name	Type
n°	m ²	m	m ³	kg	—	—
011	2.169	1.584	(3.436)	2583	VDCOMER	BRANCH(2)
013	2.0874	0.872	(1.8202)	1366	INOZZLE	BRANCH(3)
015	1.41	4.083	(5.758)	4328	DOWNCOM	ANNULUS(3)
021	(4.199)	0.638	2.679	2014	VESBOT	SINGLVOL
023	(8.4)	0.638	5.359	4029	LMIXPL	BRANCH(3)
025	(3.528)	1.013	3.574	2687	CORINPL	BRANCH(2)
031	2.5	4[0.6095]	(6.095)	4333	CORE	PIPE (4)
035	(0.808)	2.438	1.9706	1481	CORBYP	SINGLVOL
039	(3.821)	0.6085	2.3252	1600	COREX	BRANCH(3)
041	(5.692)	0.8965	5.103	3512	CORPLEN	SINGLVOL
043	(5.902)	1.584	9.349	6435	UPPLEN	BRANCH(3)
045	(5.904)	1.496	8.833	6083	DOME	BRANCH(1)
—	—	—	56.302	40453	—	—

() : data in brackets are not entered in input data , but are calculated by program from other entered data.

TABLE 4.4: SUMMARY OF REACTOR VESSEL PRESSURE DROP

Component numbers	Description		Nominal	Calculated
V153 - V111	ΔP vessel (bar)		2.165	2.156
W02502/W2302	Core bypass %		4.5	4.52
W01101/W1302+03	Upper head bypass %		0.24	0.248
V153 - V013	Inlet nozzle ΔP (bar)		0.372	0.3623
V013 - V01503	Downcomer ΔP (bar)		0.124	0.1265
V01503 - V025	Lower plenum ΔP (bar)		0.524	0.4538
V025 - V03101	Core support plate ΔP (bar)		0.2	0.197
V3101,02,03,04	Core rod friction ΔP (bar)		0.19	0.195
J3101,02,03	Core rod grids ΔP (bar)		0.4	0.398
V3104 - V041	Upper core plate ΔP (bar)		0.2	0.207
V25 - V041	Total core ΔP (bar)		1.	0.992
V041 - V111	Outlet nozzle ΔP (bar)		0.124	0.127

TABLE 4.5 : Summary of primary loop nodalisation volumes
- FIRST PRIMARY LOOP - (LOOP B)

HOT LEG

Component	Area	Length	Volume	Watermass	Name	Type
n°	m ²	m	m ³	kg	—	—
111	0.383	2.476	(0.948308)	653	HTLG 11	SNGLVOL
112	0.383	2.476	(0.948308)	653	HTLG 12	BRANCH
113 - 01	0.383	2.476	(0.948308)	653	HTLG 13	SNGLVOL
		7.428	2.844924	1959		

PUMP SUCTION LEG

131 - 01	0.487	1.806	(0.87952)	661	LSEAL B1	BRANCH
132 - 01	0.487	1.806	(0.87952)	661	B132	BRANCH
133 - 01	0.487	1.633	(0.79527)	598	LSEAL B2	PIPE
133 - 02	0.487	1.43	(0.69641)	523		
		6.675	3.25072	2443		

PUMP

140	(6.29627)	1.232	7.757	5830	PUMP B	PUMP
			7.757	5830		

COLD LEG

151 - 01	0.383	1.968	(0.754)	567	CLDLG-B1	PIPE
02	0.383	1.968	(0.754)	567		
152	0.383	1.968	(0.754)	567	CLDLG-B2	BRANCH
153	0.383	1.968	(0.754)	567	CLDLG-B3	SNGLVOL
		7.872	3 015	2268		

() : data in brackets are not entered in input data, but are calculated by program from other entered data.

TABLE 4.6 : Summary of primary loop nodalisation volumes
- SECOND PRIMARY LOOP - (LOOP A)

HOT LEG

Component	Area	Length	Volume	Watermass	Name	Type
n°	m ²	m	m ³	kg		
211- 01	0.383	2.476	(0.948308)	653	HTLGA1	PIPE
02	0.383	2.476	(0.948308)	653		
03	0.383	2.476	(0.948308)	653		
—	—	7.428	2.844924	1959	—	—

PUMP SUCTION LEG

231 - 01	0.487	1.806	(0.879522)	661		PIPE
02	0.487	1.806	(0.879522)	661		
03	0.487	1.833	(0.795271)	598		
04	0.487	1.43	(0.696410)	523		
—	—	6.675	3.250725	2443	—	—

PUMP

240	(6.296267)	1.232	7.757	5831	PUMP	PUMP
—	—	—	7.757	5831	—	—

COLD LEG

251- 01	0.383	1.968	(0.754)	567	CLDLGA1	PIPE
02	0.383	1.968	(0.754)	567		
252	0.383	1.968	(0.754)	567	CLDLGA2	BRANC
253	0.383	1.968	(0.754)	567	CLDLGA3	SNGLV
—	—	7.872	3.015	2268	—	—

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TABLE 4.7: Summary of steam generator nodalization (volumes)
PRIMARY SIDE

Component	Area	Length	Volume	Watermass	Name	Type
n°	m ²	m	m ³	kg		
121 - 221	(2.607)	1.685	4.392	3024	INBOX	BRANCH (1)
125 - 225	(2.607)	1.685	4.392	3301	OUBOX	BRANCH (1)
122/01→10 222/01→10	0.99	10x0.908	(8.989)	6436	OUTUB 1	PIPE (10)
124/01→05 224/01→05	0.99	5x1.816	(8.989)	6697	DUTUB	PIPE (5)
_____	_____	_____	26.762	19458	_____	_____

SECONDARY SIDE

612 - 712	8.048	0.735	(5.915)	4033	B612	BRANCH
613 - 713	(8.029)	0.697	5.596	4318	FEEDMIX	SINGLVOL
615/01-715/01	2.589	1.938	(5.015)	3875	SGLDCOME1	ANNULUS (4)
615/02→04 715/02→04	0.4845	3x2.550	(3x1.235)	2862	"	"
620/01→04 720/01→04	4.432	4x1.816	(4x8.0485)	11225	SG PIPE	PIPE
621 - 721	4.432	2.324	(10.300)	1766	SG RISER	BRANCH
622 - 722	4.	1.432	(5.728)	912	TOPRI1	BRANCH
611 - 711	8.048	2.424	(19.508)	5075	BYPASSB	BRANCH
623 - 723	4.	2.424	(9.696)	781	SEPARATOR	SEPARATR
625 - 725	4.	1.	(4.)	121	DRYER	BRANCH
627 - 727	8.56	1.	(8.56)	260	PLENUMB	BRANCH
630 - 730	12.56	2.093	(26.288)	797	SEDOME1	BRANCH
_____	_____	_____	136.505	36025	_____	_____

STEAM LINES

640 . 01	0.288	22.7171	(6.54252)	197	SGLB1	PIPE
02	"	"	(")	197		
03	"	"	(")	196		
650 01	0.292	32.61	(9.52212)	284	SGLB2	PIPE
02	"	"	(")	283		
03	"	"	(")	282		
740 01	0.288	22.7171	(6.54252)	197	SGLA1	PIPE
02	"	"	(")	197		
03	"	"	(")	196		
750 01	0.292	33.4646	(9.77032)	291	SGLA2	PIPE
02	"	"	(")	290		
03	"	"	(")	290		
800	0.292	13.096	(3.82403)	113	COLLECTOR	BRANCH
840	2.	10.	(20.)	583	TURBIN	BRANCH
_____	_____	_____	120.95647	3596	_____	_____

TABLE 4.8 : Summary of pressurizer nodalisation (volumes)

Component	Area	Length	Volume	Watermass	Name	Type
n°	m ²	m	m ³	kg	—	—
310	1.6964	0.3	(0.509)	302	PRES 310	BRANCH
311	1.6964	0.6	(1.018)	604	PRES 311	BRANCH
312	2.5164	1.766	(4.444)	2637	PRES 312	BRANCH
313	2.5164	1.766	(4.444)	2638	313	BRANCH
314	2.5164	1.539	(3.8727)	2299	314	BRANCH
315	2.5164	1.515	(3.8123)	389	315	BRANCH
316	2.5164	0.8	(2.0131)	205	316	BRANCH
317	2.5164	0.6	(1.5098)	154	317	BRANCH
320-01	2.5164	0.4	(1.0066)	103	PRES 320	PIPE
02	1.6964	0.3	(0.50892)	52		
03	1.6964	0.2	(0.33928)	35		
04	1.6964	0.15	(0.25446)	26		
05	1.6964	0.10	(0.16964)	17		
06	1.6964	0.10	(0.16964)	17		
330	1.6964	0.05	(0.08482)	9	PRES DCM	BRANCH
—	—	—	24.15626	9487	—	—

SURGE LINE

301-01	0.03875	6.3465	(0.24593)	169	SURGELN	PIPE
-02	0.03875	6.3465	(0.24593)	166		
-03	0.03875	4.07	(0.15771)	103		
—	—	—	0.64957	438	—	—

() : data in brackets are not entered in input data, but are calculated by program from other entered data.

STATE OF TEXAS, COUNTY OF []

No.	Name	Address	City	County	State
1	John W. Smith	123 Main St	Houston	Harris	Texas
2	Jane D. Doe	456 Elm St	Dallas	Dallas	Texas
3	Robert E. Brown	789 Oak St	Austin	Cook	Texas
4	William F. Green	101 Pine St	San Antonio	Brewster	Texas
5	Elizabeth C. White	202 Cedar St	Fort Worth	Tarrant	Texas
6	James H. Black	303 Birch St	El Paso	El Paso	Texas
7	Mary K. Gray	404 Walnut St	San Diego	San Diego	Texas
8	Charles L. Pink	505 Spruce St	Phoenix	Maricopa	Texas
9	Patricia A. Blue	606 Ash St	Portland	Clatsop	Texas
10	Richard M. Yellow	707 Hickory St	Seattle	King	Texas
11	Susan B. Purple	808 Maple St	Denver	Denver	Texas
12	Thomas G. Red	909 Poplar St	Chicago	Cook	Texas
13	Linda S. Green	1010 Chestnut St	Los Angeles	Los Angeles	Texas
14	Michael P. Blue	1111 Olive St	New York	New York	Texas
15	Barbara L. Yellow	1212 Broadway St	San Francisco	San Francisco	Texas
16	David R. Purple	1313 Market St	San Jose	San Jose	Texas
17	Jennifer M. Red	1414 Mission St	San Jose	San Jose	Texas
18	Christopher J. Blue	1515 Folsom St	San Francisco	San Francisco	Texas
19	Amanda N. Yellow	1616 Divisadero St	San Francisco	San Francisco	Texas
20	Gregory K. Purple	1717 Geary St	San Francisco	San Francisco	Texas
21	Stephanie H. Red	1818 Sutter St	San Francisco	San Francisco	Texas
22	Brandon T. Blue	1919 Powell St	San Francisco	San Francisco	Texas
23	Nicole M. Yellow	2020 Stockton St	San Francisco	San Francisco	Texas
24	Kevin L. Purple	2121 Vallejo St	San Francisco	San Francisco	Texas
25	Michelle A. Red	2222 Lombard St	San Francisco	San Francisco	Texas
26	Timothy J. Blue	2323 Broadway St	San Francisco	San Francisco	Texas
27	Christina B. Yellow	2424 Market St	San Francisco	San Francisco	Texas
28	Jonathan D. Purple	2525 Mission St	San Francisco	San Francisco	Texas
29	Sarah E. Red	2626 Folsom St	San Francisco	San Francisco	Texas
30	Matthew F. Blue	2727 Divisadero St	San Francisco	San Francisco	Texas
31	Olivia G. Yellow	2828 Geary St	San Francisco	San Francisco	Texas
32	Isaac H. Purple	2929 Sutter St	San Francisco	San Francisco	Texas
33	Grace I. Red	3030 Powell St	San Francisco	San Francisco	Texas
34	Henry J. Blue	3131 Stockton St	San Francisco	San Francisco	Texas
35	Abigail K. Yellow	3232 Vallejo St	San Francisco	San Francisco	Texas
36	Lucas L. Purple	3333 Lombard St	San Francisco	San Francisco	Texas
37	Chloe M. Red	3434 Broadway St	San Francisco	San Francisco	Texas
38	Leo N. Blue	3535 Market St	San Francisco	San Francisco	Texas
39	Penelope O. Yellow	3636 Mission St	San Francisco	San Francisco	Texas
40	Julian P. Purple	3737 Folsom St	San Francisco	San Francisco	Texas
41	Skylar Q. Red	3838 Divisadero St	San Francisco	San Francisco	Texas
42	Wyatt R. Blue	3939 Geary St	San Francisco	San Francisco	Texas
43	Zoe S. Yellow	4040 Sutter St	San Francisco	San Francisco	Texas
44	Easton T. Purple	4141 Powell St	San Francisco	San Francisco	Texas
45	Madelyn U. Red	4242 Stockton St	San Francisco	San Francisco	Texas
46	Lincoln V. Blue	4343 Vallejo St	San Francisco	San Francisco	Texas
47	Alaina W. Yellow	4444 Lombard St	San Francisco	San Francisco	Texas
48	Grayson X. Purple	4545 Broadway St	San Francisco	San Francisco	Texas
49	Isabella Y. Red	4646 Market St	San Francisco	San Francisco	Texas
50	Julian Z. Blue	4747 Mission St	San Francisco	San Francisco	Texas

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80	Matthew F. Blue	2727 Divisadero St	San Francisco	San Francisco	Texas
81	Olivia G. Yellow	2828 Geary St	San Francisco	San Francisco	Texas
82	Isaac H. Purple	2929 Sutter St	San Francisco	San Francisco	Texas
83	Grace I. Red	3030 Powell St	San Francisco	San Francisco	Texas
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85	Abigail K. Yellow	3232 Vallejo St	San Francisco	San Francisco	Texas
86	Lucas L. Purple	3333 Lombard St	San Francisco	San Francisco	Texas
87	Chloe M. Red	3434 Broadway St	San Francisco	San Francisco	Texas
88	Leo N. Blue	3535 Market St	San Francisco	San Francisco	Texas
89	Penelope O. Yellow	3636 Mission St	San Francisco	San Francisco	Texas
90	Julian P. Purple	3737 Folsom St	San Francisco	San Francisco	Texas
91	Skylar Q. Red	3838 Divisadero St	San Francisco	San Francisco	Texas
92	Wyatt R. Blue	3939 Geary St	San Francisco	San Francisco	Texas
93	Zoe S. Yellow	4040 Sutter St	San Francisco	San Francisco	Texas
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95	Madelyn U. Red	4242 Stockton St	San Francisco	San Francisco	Texas
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97	Alaina W. Yellow	4444 Lombard St	San Francisco	San Francisco	Texas
98	Grayson X. Purple	4545 Broadway St	San Francisco	San Francisco	Texas
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100	Julian Z. Blue	4747 Mission St	San Francisco	San Francisco	Texas

STATE OF TEXAS, COUNTY OF []

FIG 4.2 DOEL 1/2 REACTOR VESSEL NODALISATION

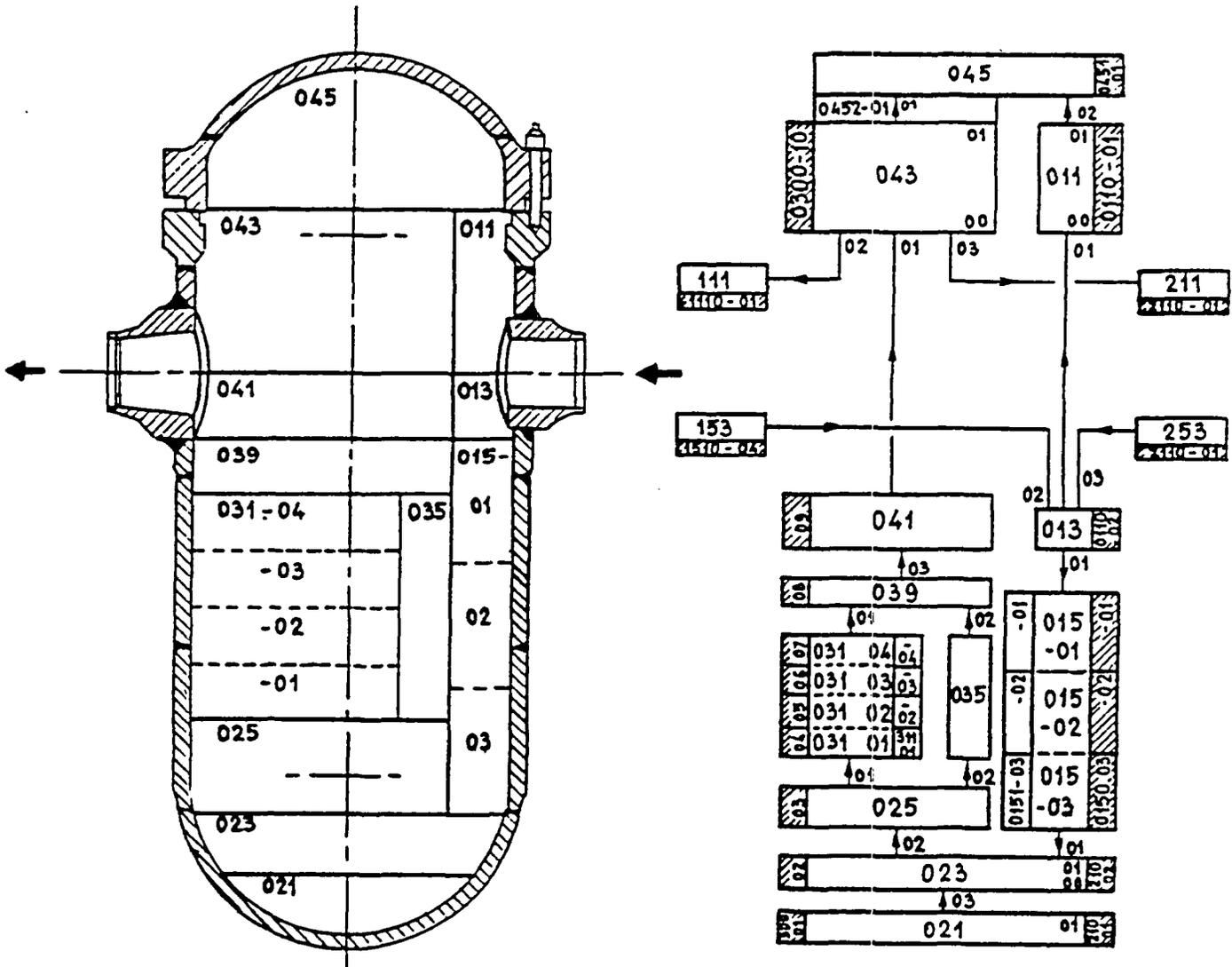


FIG. 4.3 DOEL 1/2 STEAM GENERATOR NODALISATION

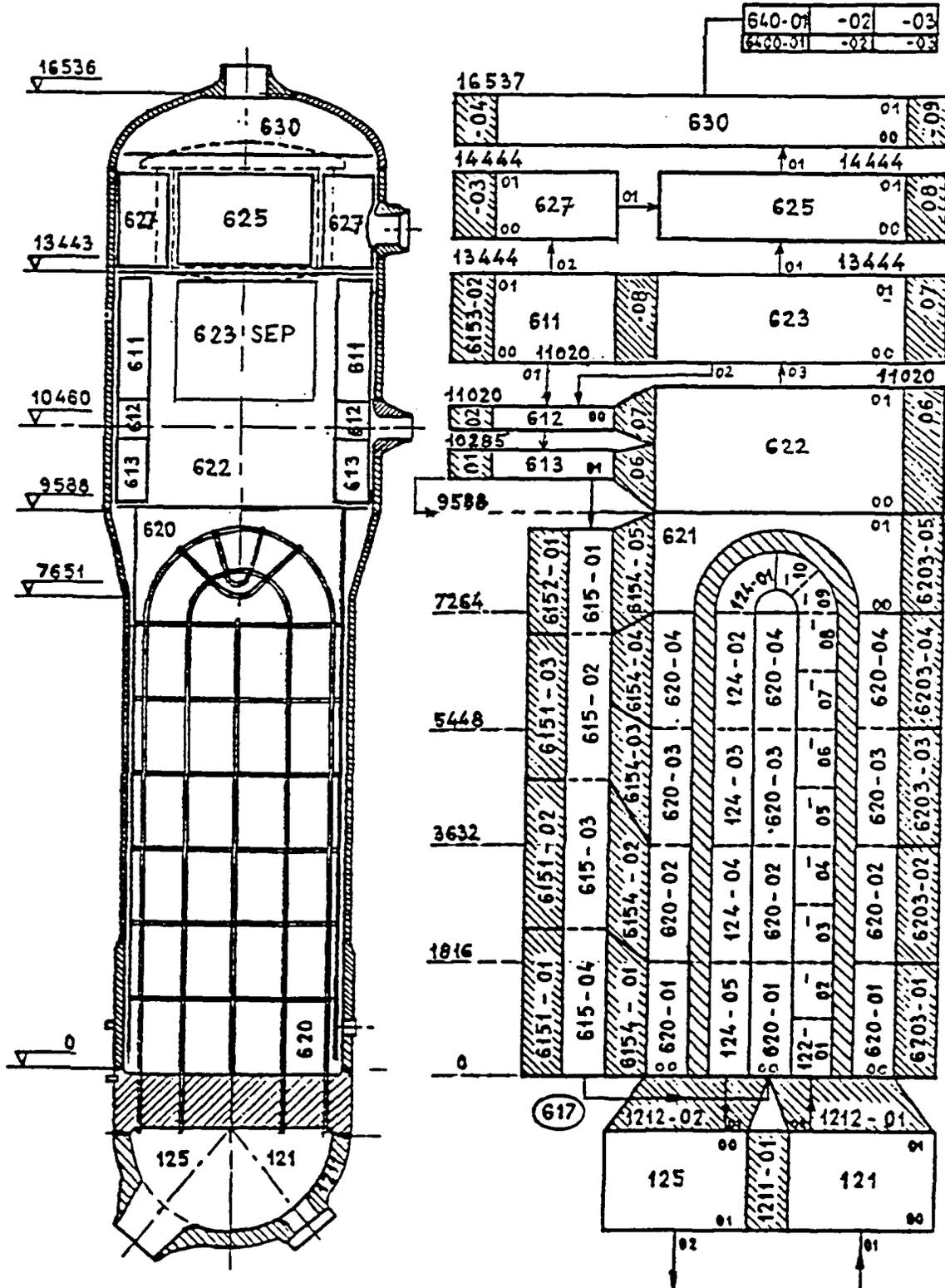


FIG4.4 PRESSURIZER LEVEL

- The level programmer is linear between given limits *

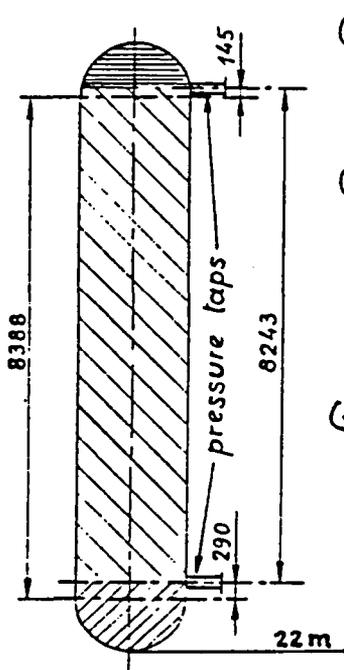
Minimum level for a no-load

T_{avg} of 283.4°C (556.55°K) 24.65% of level span.

Maximum level for a full load

T_{avg} of 299.6°C (572.75°K) 58% of level span.

- Level span **



(01) volume  (below the pressure tap)

$$V = 1.5268 + 0.7297 = 2.2565 \text{ m}^3$$

(02) volume  (between the pressure taps)

The section area variation along the 145mm has been neglected.

$$V = 8.243 \times 2.5164 = 20.74346 \text{ m}^3$$

(03) volume  (above the pressure tap)

$$V = 1.16192 \text{ m}^3$$

$$\text{volume surge line } V = 0.64957 \text{ m}^3$$

The CONTROL VARIABLE 900 gives the collapsed water level particular values.

(- 0.109) indicates that only the pressurizer is empty

(- 0.14) indicates that both the pressurizer and the surge line are empty.

(+ 1.056) water solid conditions

* Set point study for the DOEL nuclear power station (WENX 73/49, november 30, 1973)

** Drawing COCKERILL 4306/9 , 07.03.69, rev. "K" 27.03.72 .

RELATIONSHIP BETWEEN REAL COLLAPSED WATER LEVEL AND MEASURED LEVEL BY :
 LEFT : COLD CALIBRATED ΔP CELL RIGHT : HOT CALIBRATED ΔP CELL

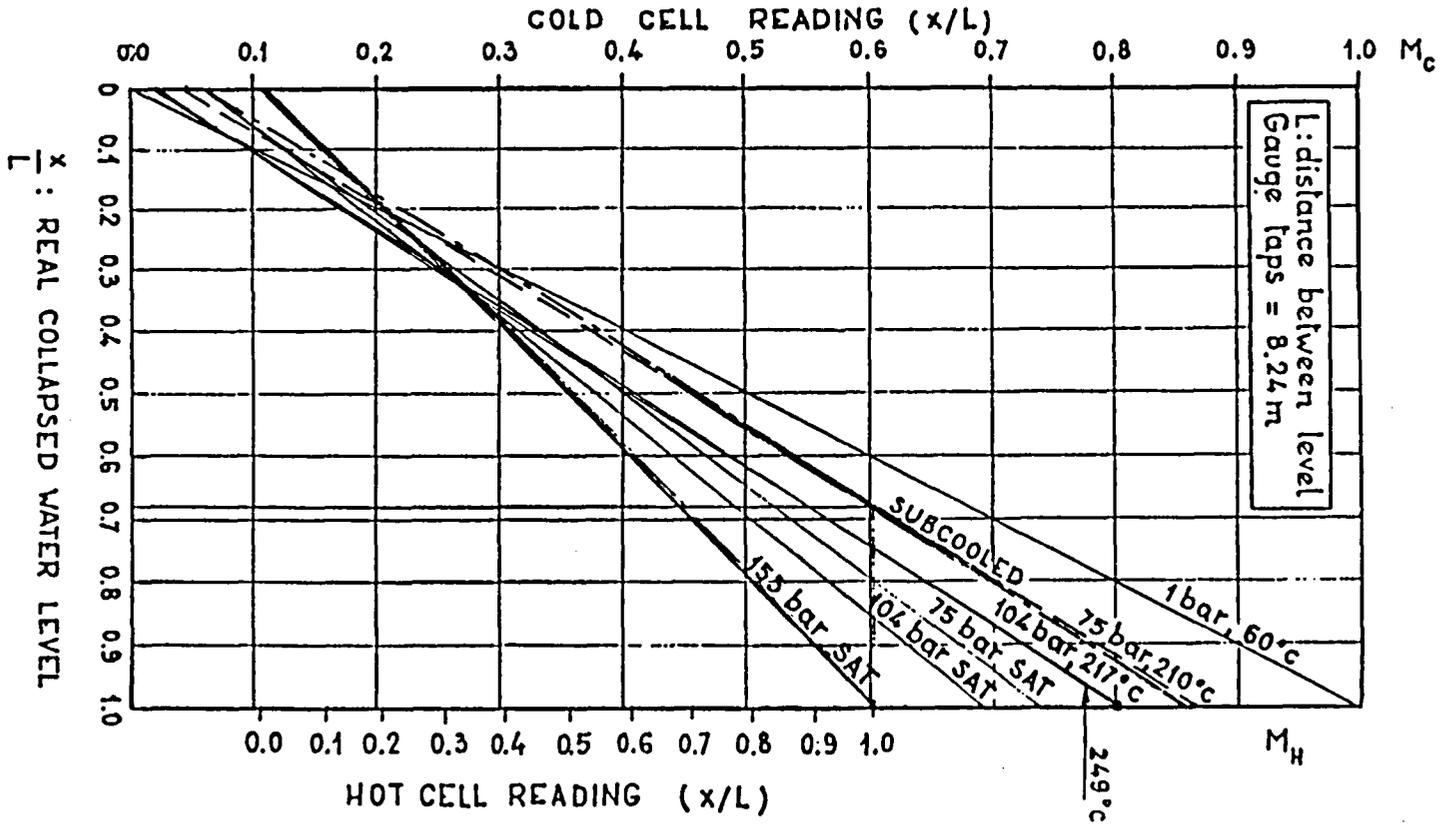
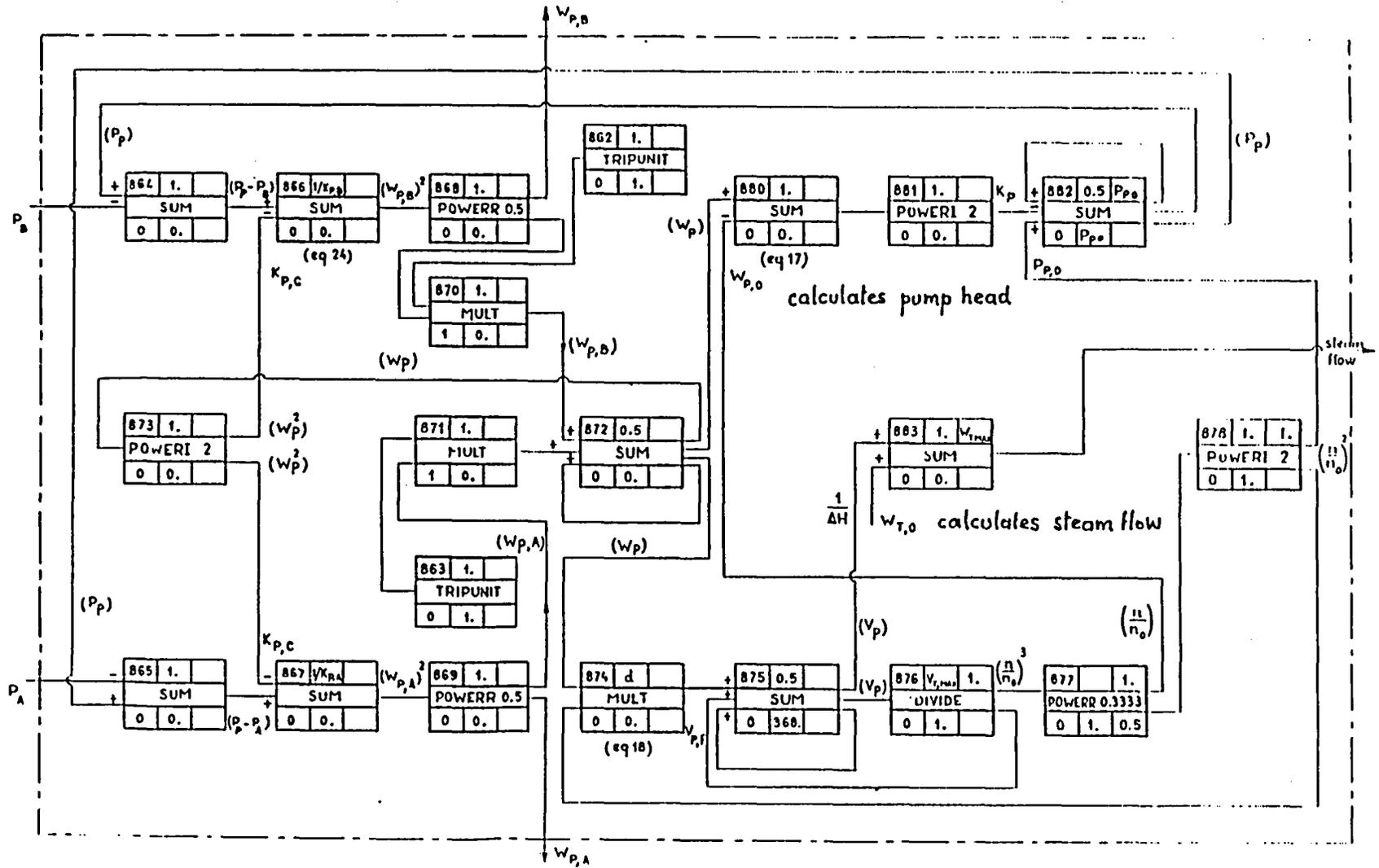


FIG 4.5

FIG. 4.6 TURBOPUMP CONTROL SYSTEM (REV. A)



note: - 1) The values for scale factor (s) and multiplier constants (Ai) are given in table I-2
 2) Values in parenthesis () are symbols of intermediate parameters for illustration only (are not multipliers)

5. BASE CASE RESULTS

The microfiches joined to this report contain the complete output file for this case.

The results are discussed on the basis of an exhaustive set of figures divided in several groups.

The numbering logic for these figures is as follows :

group 2 : pressuriser pressure
group 3 : pressuriser level
group 4 : primary coolant pressure
group 5 : primary coolant temperatures
group 6 : mass addition or depletion from primary system
group 7 : primary system energy parameters
group 8 : mass flow rates in RCS
group 9 : miscellaneous for RCS

group 10 : pressures in SG's
group 11 : water levels in the SG's
group 12 : temperatures in the SG's
group 14 : mass addition or depletion from SG's
group 15 : mass evolution in SG's

To be consistent with the numbering logic of the figures in this report, the chapter number (5) will precede the figure number.

The main results of the base case will be described on the basis of these figures, whereby apparent anomalies will be discussed in detail in chapter 6.

. fig. 5.2.1. Pressure evolution in the pressuriser

For the initial depressurisation period (up to 900 s), the calculated pressure drop agrees well with the measured pressure drop (reconstructed from fig. 4 in annexe 1). When cooldown starts at 900, the depressurisation rate increases down to a pressure of 107 bar whereby the HPSI system stops the depressurisation slightly below the HPSI cut-off head of 108 bar, and compares to a measured plateau pressure of 107 bar (figure 3 in annexe 1).

When the operator actuated full spray at 2106 s elapsed time, a sharp pressure drop is calculated and observed, followed by

a repressurisation when the spray is stopped. The kinks in the repressurisation are probably caused by anomalous vapour condensation (or water evaporation) whenever the water level crosses a volume boundary in the pressuriser.

. Fig 5.3.1. and Fig. 5.3.2. Pressuriser water level.

The curves shown are respectively :

. The recorded water level (LRCA - PR11) in dash-dot line

. The collapsed water level $L_c = \left[\sum_i \alpha_i V_i - V_b \right] / \Delta \gamma$ (CNTRLVAR 900)

. wherein V_i is the volume of component i of pressuriser

V_b is the volume below the lower level tap (fig.4.4)

$\Delta \gamma$ is the pressuriser volume between level taps
(fig.4.4)

. The hot ΔP cell level as calculated from expression b in paragraph 4.1.5.

. The cold ΔP cell level as calculated from expression a in par. 4.1.5.

The collapsed water level drops below the 0 % water level, meaning that the pressuriser drains below the lower level tap, down to the level of the hot leg connection.

The hot cell level, which is a ΔP measured with a ΔP cell calibrated at 100 % nominal condition, follows closely the recorded water level during initial depressurization, whereby the 0 level is reached at about 500 s. This close agreement confirms that the leak flowrate, estimated from a water level increase in the affected S.G., and the postulated charging minus letdown flow, are close to the plant values.

The cold calibrated water level measurement cell still records a finite level since it measures the normalised weight of the vapour in the pressuriser (see CONTROL VARIABLE 902).

During the period of pressuriser spray, the pressuriser fills up with water which is about 150°C colder than the temperature at which the ΔP level cells were calibrated.

From the calibration curves, shown in figure 4.5, one observes that for fluid temperatures in the pressuriser around 210°C, a hot cell reading of 100 % corresponds to a collapsed water level of about 68%, which in turn would show a reading of 60 % on the cold calibrated gauge. These were also the values which were calculated in the very first simulation study (by means of RELAP-5 MOD-1). As will be seen further, the calculated cooldown is larger, resulting in a stronger depressurization during spray, and a faster fill-up of the pressuriser.

At 2500 s, the plant recorded level, the hot cell level and also the collapsed water level reach about 100 %, while the cold gauge indicates 80% water level. These calculated data are consistent for an average fluid density in the pressuriser of 800 kg/m³ corresponding to an average fluid temperature of 249°C (522°K) (the fluid temperature in the pressuriser varies between 473°K and 588°K).

The calibration line for 249°C has been added in fig. 4.5.

. Fig. 5.3.2. compares the measured and calculated pressuriser level by means of the hot calibrated ΔP cell (the only two quantities that merit exact comparison). The agreement is good during the initial depressurization. For the remainder of the transient no conclusion can be drawn since the values are limited artificially between 0 and 100 %.

. Fig. 5.4.1. pressure evolution in hot leg A.

The calculated pressure drop agrees very well with the recorded pressure evolution except for two periods, respectively at the initiation of the cooldown (around 1000 s) and during the pressuriser spray period (between 2106 and 2287 seconds).

Since the exact timing and opening rate of the intact SG atmospheric steam relief valve are unknown (see also chapter 6.1) the discrepancy at the initiation of the cooldown must be attributed to unprecise boundary condition rather than code errors.

The excessive depressurization during the pressuriser spray activation is caused by excessive cooldown of the RCS as will be further discussed in chapter 6.2.

Figures 5.5.2. till 5.5.9. : Temperature evolution in the primary loops

figures 5.5.4, 5.5.5., 5.5.6 and 5.5.7., for which corresponding recordings exist, all manifest too strong a cooldown of the RCS by the intact steam generator, and will be discussed in chapter 6.2.

The temperature discontinuity at the time of the trip of the main coolant pump in the affected loop (at 1112 s), visible in figures 5.5.7 and 5.5.8, results from a flow reversal in the loop, whereby cold HPSI water, injected in the cold legs, is now redirected towards the pump and influences the temperature sensor (2RC29) located between the HPSI location and the pump.

This sudden temperature drop is not noticeable in the recordings. This may be caused by injection of all HPSI in both cold legs, while in the plant at least one injection line to the downcomer was open.

Fig. 5.5.9. illustrates clearly the increase in the vapour temperature at the pressuriser dome between 1200 s and initiation of pressuriser spray, due to structural heat release from the pressuriser walls to vapour, leading to a superheat of about 9°C. This structural heat compensates the vapour condensation at the steam/water interface and maintains a high pressuriser pressure, during the SI phase.

During the spraying period, the pressuriser temperature is following the saturation temperature. When the spraying is

stopped, a water piston effect is seen whereby the vapour is compressed adiabatically to about 40°C superheat at which point the heat losses to the structure surrounding the pressuriser dome (VOL 320), leads to a final reduction in the vapour temperature from 635°K down to 622°K.

This figure may illustrate why the simulation of the structural heat of the pressuriser is important in order to understand the real pressuriser behaviour, and may explain why the pressuriser cannot be filled up due to the HPSI injection when heat is released from the structure to the vapour in an empty pressuriser, until spray is initiated. A volumetric balance shows that the HPSI water injected in the RCS compensates the primary coolant shrinking without increasing the water level in the pressuriser, as can be seen also from figs 5.3.1. and 5.8.3.

. The figures of group 5.6. illustrate various RCS inflow and outflow quantities.

Figures 5.6.1 and 5.6.2. illustrate the evolution of the break mass flow rate. While fig. 5.6.2. shows the break flow rate as calculated, fig. 5.6.1. shows the result of a integration differentiation operation intended to smooth out the calculated flowrate when the value oscillates from one cycle to the next (which was observed in a former calculation by means of the code RELAP-5 MOD-1). Since no such oscillation are present, which is a remarkable improvement in MOD-2, both figures yield essentially identical results except close to time zero.

The mass flowrate clearly follows the pressure evolution except for a gradual increase between 1500 s and 1900 s due to a decrease in the affected SG pressure (see fig. 5.10.1), since the junction flowrate is not choked during this transient.

Fig. 5.6.5. shows the charging and the letdown flow rates. While the letdown closes at 41 s due to low pressuriser level,

the charging flow increases in order to maintain the pressuriser level. Since the breakflow exceeds the charging flow a second pump is started at 170 s, and the third charging pump starts at 900 s.

Fig. 5.6.6 illustrates the HPSI mass flow rate. With a compensation logic for the HPSI flow (see chapter 4.2.2) relative strong fluctuations are observed due to minor variations of the back pressure (fig. 5.4.1). For future runs, the lag due to the compensation, should be reduced. Combining all inflow and outflow from the primary system, one obtains figure 5.6.3, illustrating a net coolant loss until the third charging pump was started around 900 s (when pressuriser was empty), and a net inventory increase when the HPSI pumps start injecting at 1200 s.

. Fig. 5.6.10. illustrates the variation of the primary coolant inventory (integral of fig. 5.6.3).

. Figures of group 5.7 illustrate various energy fluxes in the primary system.

The core thermal power is shown in fig. 7.1. The increase from 6 MWth to about 7 MWth between 900 and 1500 sec. results from the cooldown initiated via intact SG. The power spike at 2120 s coincides with the start of the second coolant pump whereby colder water ($\Delta T = 5^\circ\text{C}$), accumulated in the cold leg of affected loop, reverses direction and is forced through the core.

The energy removed by the break is shown in figure 5.7.2.

. Fig. 5.7.3. illustrates the energy transferred from the primary to each steam generator (Heat input in the SG riser volume).

This figure illustrates many important phenomena (physical and numerical) which dominate the RCS behaviour.

In general, one observes that hardly any heat is transferred via the SG's until the cooldown is initiated at 900 s. The intact SG acts as a heat sink during the remainder of the transient. The cooling power increases during the opening of the SG relief valve (900 - 1200 s), again at 1428 when the SG moto pumps, and at 1482 when the turbine driven auxiliary feedwater pump start injecting cold auxiliary feedwater in the intact SG.

The affected SG becomes a heat source which explains the asymmetric behaviour of a plant under SGTR conditions. Even when the turbine driven auxiliary feedwater pump is started at 1482 s, the affected SG acts as a heat sink (contrary to expectation).

The sudden power reversal to the affected SG at the instant the turbine driven auxiliary feedwater pump is tripped (1900 s) is caused by an anomalous SG behaviour and will be explained in chapter 6.3.

The very large heat extraction by the affected SG between 1900 and 2100 s reinforces the excessive cooldown of the primary system enhancing the discrepancies observed in the primary cooldown rate and the ensuing excessive depressurization during the spray period (e.g. fig. 5.4.1, 5.5.5, 5.5.8).

. Fig. 5.7.4. expresses the heat flux extracted from the SG tubes (primary side) and is basically identical to fig. 5.7.3 except for the sign, the structural heat in the SG tube bundle and possible heat exchanges between riser and surroundings.

. Fig. 5.7.5. illustrates the net energy balance in the primary system, and shows that the net energy balance is mostly negative (leading to a sustained depressurization) except during the spray period which is caused by the very large injection of HPSI water, with small but non negligible

energy content. The large spike around 1900 s is caused by the sudden energy extraction by the affected SG (see chapter 6.3).

. Fig. 5.7.6. illustrates the structural heat input respectively in the pressuriser and the total primary system.

It is important to note that the core residual heat is about 6MW and at the end of the cooldown period, the structural heat input (before spray) also reaches a value of 6 MW, hence illustrating the importance of modelling structural heat release.

The sudden increase in structural heat release when spray is actuated results from a strong cooldown of the pressuriser shell in contact with cold water, combined with the actuation of the pressuriser heaters at 2230s.

. Figures of group 5.8 illustrate flowrates internal in the reactor coolant systems such as the loop flowrate (fig. 5.8.1.), the surgeline mass flow rate (fig. 5.8.3) and the pressuriser spray flow (fig. 5.8.5.).

The surgeline mass flow rate remains negative, indicating pressuriser emptying until the HPSI is actuated at 1200 s. However, the filling of the pressuriser does not start until the spray is actuated.

The miscellaneous figures of group 5.9. illustrate :

. the subcooling in the primary system (fig. 5.9.2.) :

The initial subcooling is very high corresponding to close to hot standby condition (about 90°C), and decreases due to the RCS depressurization. When cold SI water is injected, the pressure is stabilised while the cooldown continues leading to an increase in the subcooling.

The anomalous strong cooldown at 1900 s is caused by the excessive heat transfer to the affected SG (fig. 5.7.3.) and

causes a sudden increase in the subcooling around 1940 s. The subcooling reaches a constant value around 108°C, until a sharp drop occurs during the spray period; and increases again when the spray is stopped.

- . status plot of downcomer fluid temperature versus downcomer pressure (fig. 5.9.3.).

On same plot is shown the limiting vessel integrity curve (Reference temperature for nil ductility transition) for a 40 year plant condition. This illustrates the PTS (Pressurized thermal shock) problem potential of such transient.

- The integral mass balance in the primary system is shown in fig. 5.9.9. (Relative values)

Figures of group 5.10 illustrate the pressure evolution in both steam generators.

For the intact S.G. (fig. 5.10.2), while the calculated depressurization rate is acceptable, there is a constant lag between measured and calculated pressure. This discrepancy may be attributed to misjudgment in the cooldown initiation time, or a time-shift problem in the recorded curves.

For the affected S.G. (fig. 5.10.3), several features show up :

- . the calculated pressurization until 1482 s, is too high.
- . When the auxiliary feedwater is initiated at 1482 s, steam taken from this SG to drive the turbine driven auxiliary feedwater pump creates a depressurization with a rate in good agreement with the observed rate.
- . At 1710 sec, the vapour void fraction in volume 627 drops below 60 % and strong vapour condensation sets in producing a steeper pressure drop, until the turbine driven auxiliary feedwater pump is stopped at 1900s.
- . The pressure anomalies observed after 1900 s are caused by condensation problems which will be discussed in chapter 6.3.

. Figures of group 5.11 illustrate different water levels in both steam generators.

Broadband collapsed water levels in the downcomer and the riser are shown in figure 11.1.

Since no broadband measurements were recorded at the plant, the plant comparison data in figures 5.11.6 and 5.11.7 are obtained by adding the distance between tube-plate and lower level tap of the narrow-band level measurement.

No strict comparison between plant data and calculated data should be attempted for these figures.

. Comparison of measured and calculated water level in the intact S.G. is shown in figure 5.11.8. When the cooldown is started by opening the atmospheric steam relief valve between 900 and 1200 s, an excessive level swell is observed in the riser (see also fig. 5.11.1) leading to a hold up of the water in the downcomer and the region located at the small band level measurement taps. This leads to an initial offset in the calculated water level followed by a level drop with a gradient which agrees well with the measured level gradient (see also chapter 6.2).

. Figure 5.11.9. illustrates the comparison between calculated and measured water level in the affected S.G. Since the leak flow rated was tuned to the level rise, good agreement is thus built-in as input to the code.

. Figures of group 5.12 illustrate temperature evolution in the steam generators

The steam temperatures in the SG domes reflect the pressure evolution in the steam generators (close to saturation).

. Figure 5.12.2. illustrates the evolution of the steam generator inlet and outlet (primary side) and the SGB riser, temperature.

When the B loop coolant pump is stopped at 1112 sec, flow reversal in this loop causes a temperature inversion at SG inlet and outlet (inlet temperature above outlet temperature) since the affected SG acts as a heat source.

When the auxiliary feedwater motor-driven pump is stopped (1900 s), cold water hanging up in the SGB downcomer, enters the riser leading to a temporary very strong cooldown of the riser region and a strong drop in the primary SG inlet temperature (see also fig. 5.7.3.).

When the main coolant pump is started at 2105 s, the temperature inversion (inlet-outlet) is reversed.

. Fig. 5.12.3. illustrates the evolution of the temperatures in the intact SGA (primary inlet, outlet and SG bundle region). The closeness of the temperatures illustrates the high heat transfer efficiency during cooldown.

. Finally, fig. 5.12.4. gives an illustration of the strong fluid temperature stratification in the bottled-up affected S.G. after 2400 s. (see also figure 5.12.1 for dome vapour temperature).

The large temperature drop in the bottom of the riser around 1900 sec. is another illustration of the sudden cooldown when cold water, accumulated in the SG downcomer, suddenly falls down and cools the SG bundle region (see also chapter 6.2). This figure may give an indication of the challenges to the code to evaluate the correct flow and heat transfer regime under such conditions.

Figures of group 5.14 illustrate various mass addition and release rates from both steam generators.

. Figure 5.15.2 illustrates the net mass addition and depletion rates for both SG's.

. Figure 5.15.1 illustrates the total mass evolution in both SG's.

In conclusion one could state that the code calculation does present the overall macroscopic behaviour in primary and secondary system, except for three areas where large discrepancies are observed. These areas will be discussed in next chapter.

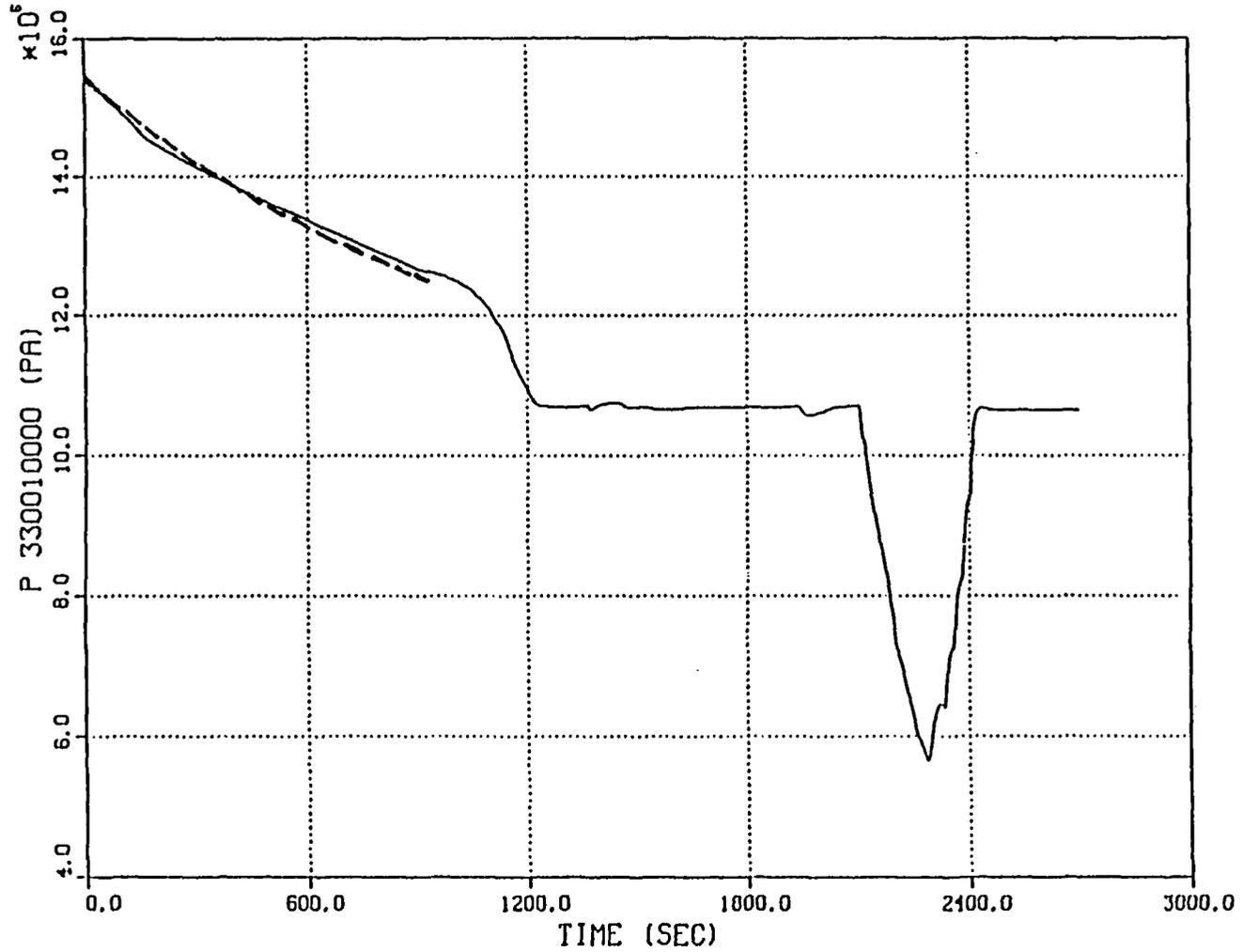
The following main phenomena, observed in the plant, are also simulated well by the code :

- . Non-equilibrium pressuriser behaviour in voided and partially subcooled conditions.
- . Counter current flow in the pressuriser during the spray period result in acceptable coolant distribution and depressurisation, in contrast to the RELAP-5 MOD-1 code (Ref. 2). Former calculations with RELAP-5 MOD-1 required artificial external steam recirculation to avoid hang-up of the spray water at top of pressuriser.
- . Structural heat release to the voided pressuriser during cooldown leads to a situation whereby the water inventory in the pressuriser remains low, even when 4 HPSI pumps are operational.
- . The cooldown gradient between 1200 s (fully opened relief valve) and start of the auxiliary feedwater system (1428 s) in the RCS matches with the plant cooldown gradient, which illustrates a correct heat transfer between RCS and both SG's (in direct and reverse direction).

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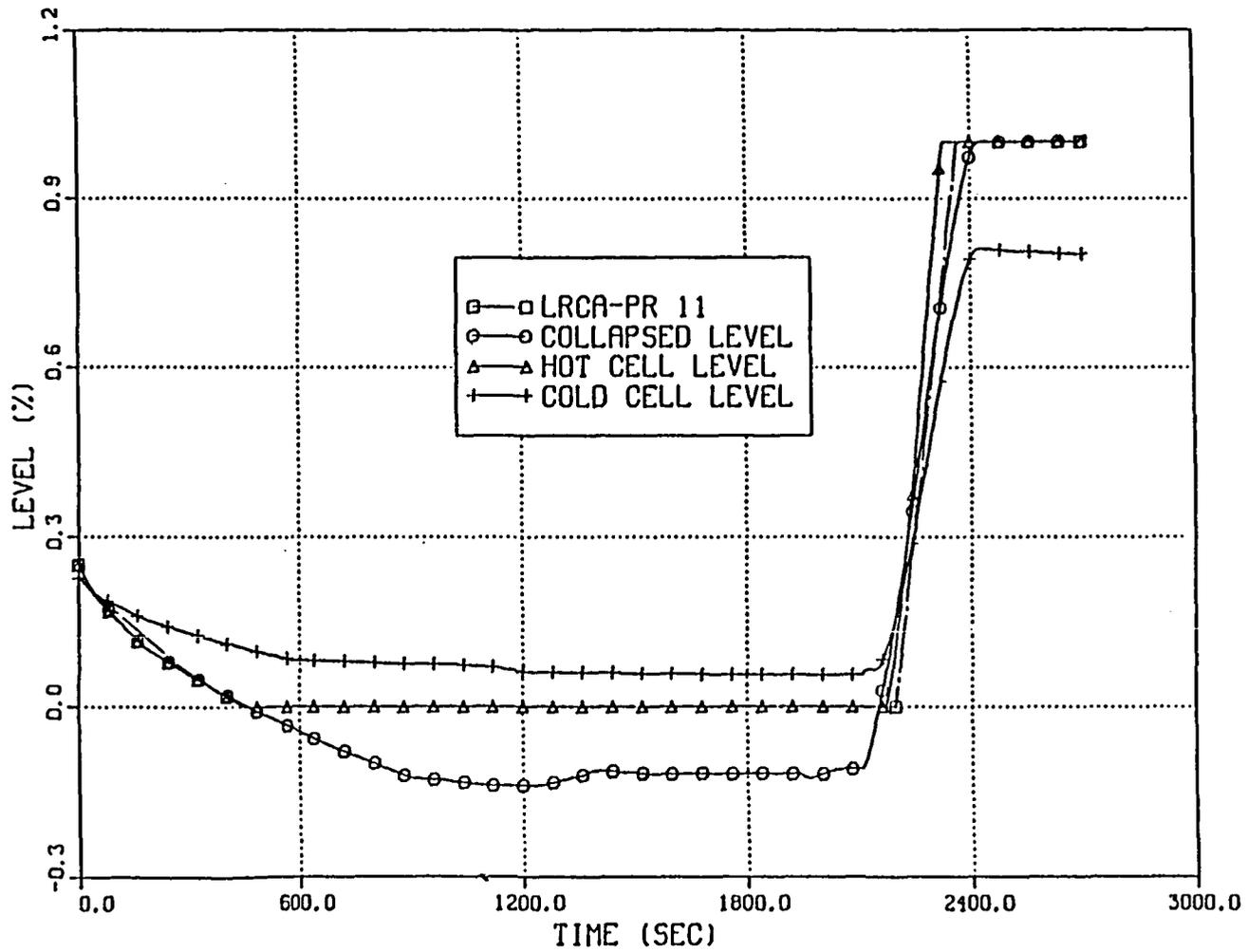
5. FIG 2.1 PRESSURE EVOLUTION IN PRESSURIZER



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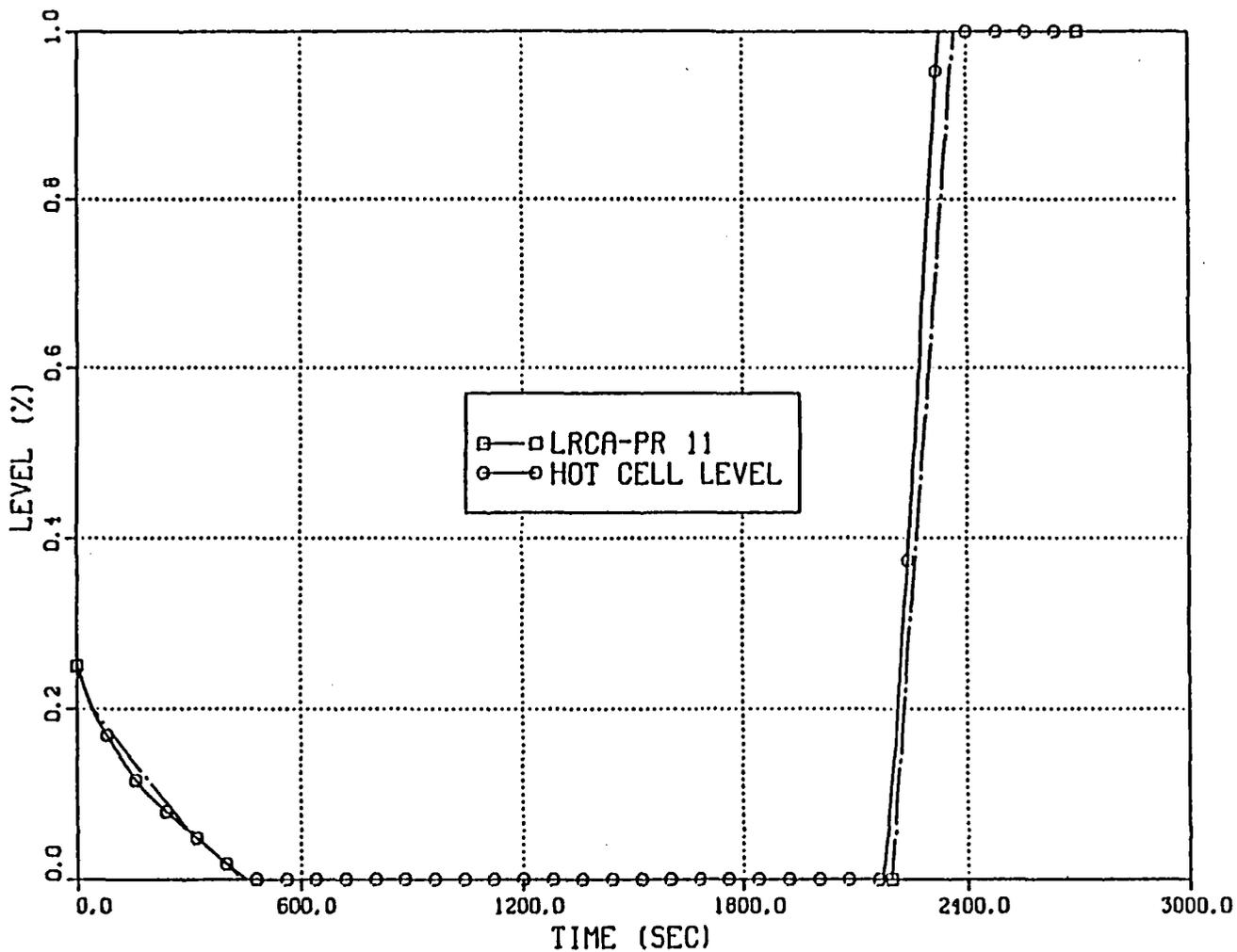
5. FIG 3.1 WATER LEVEL IN PRESSURIZER



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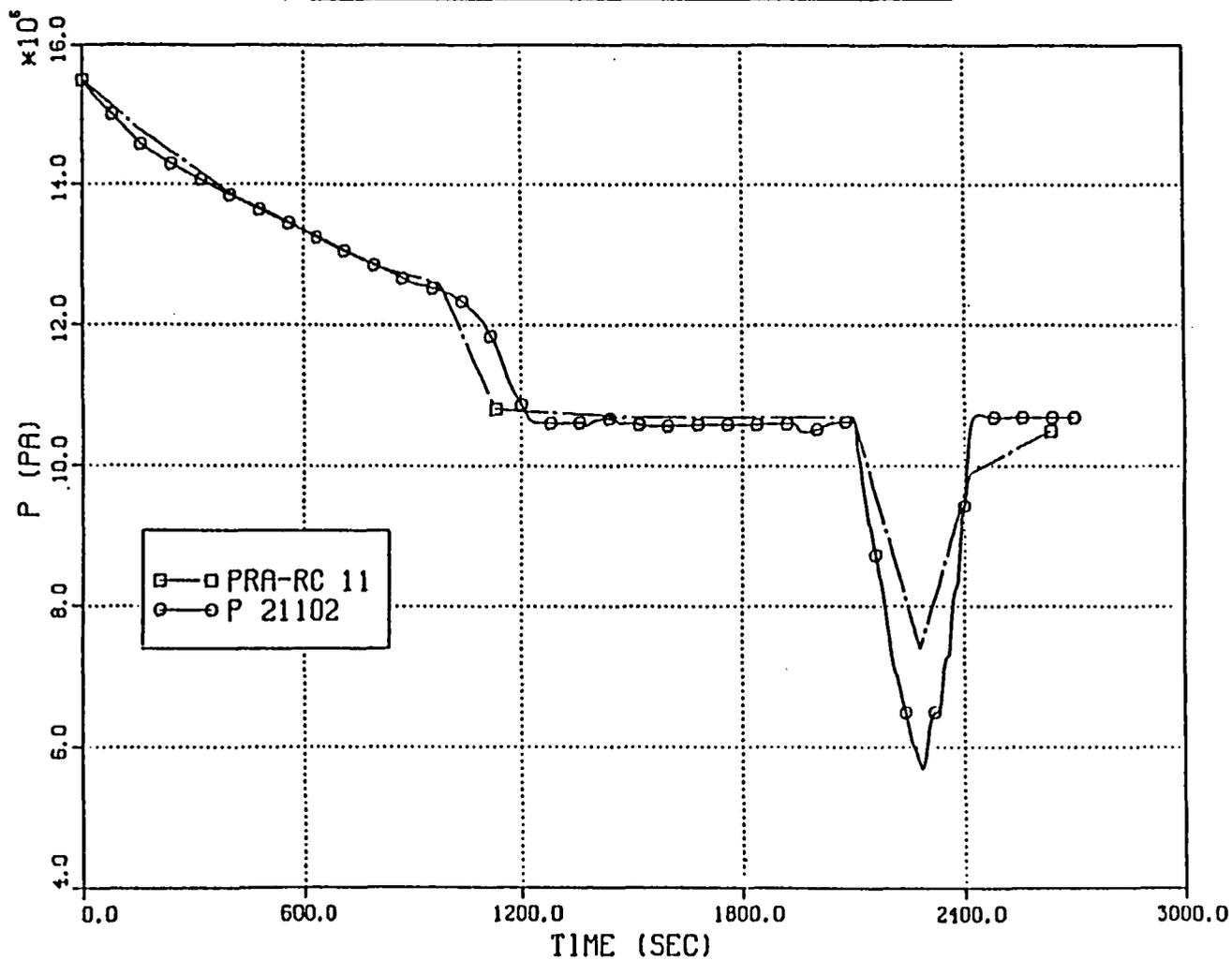
5. FIG 3.2 MEASURED WATER LEVEL IN PRESSURIZER



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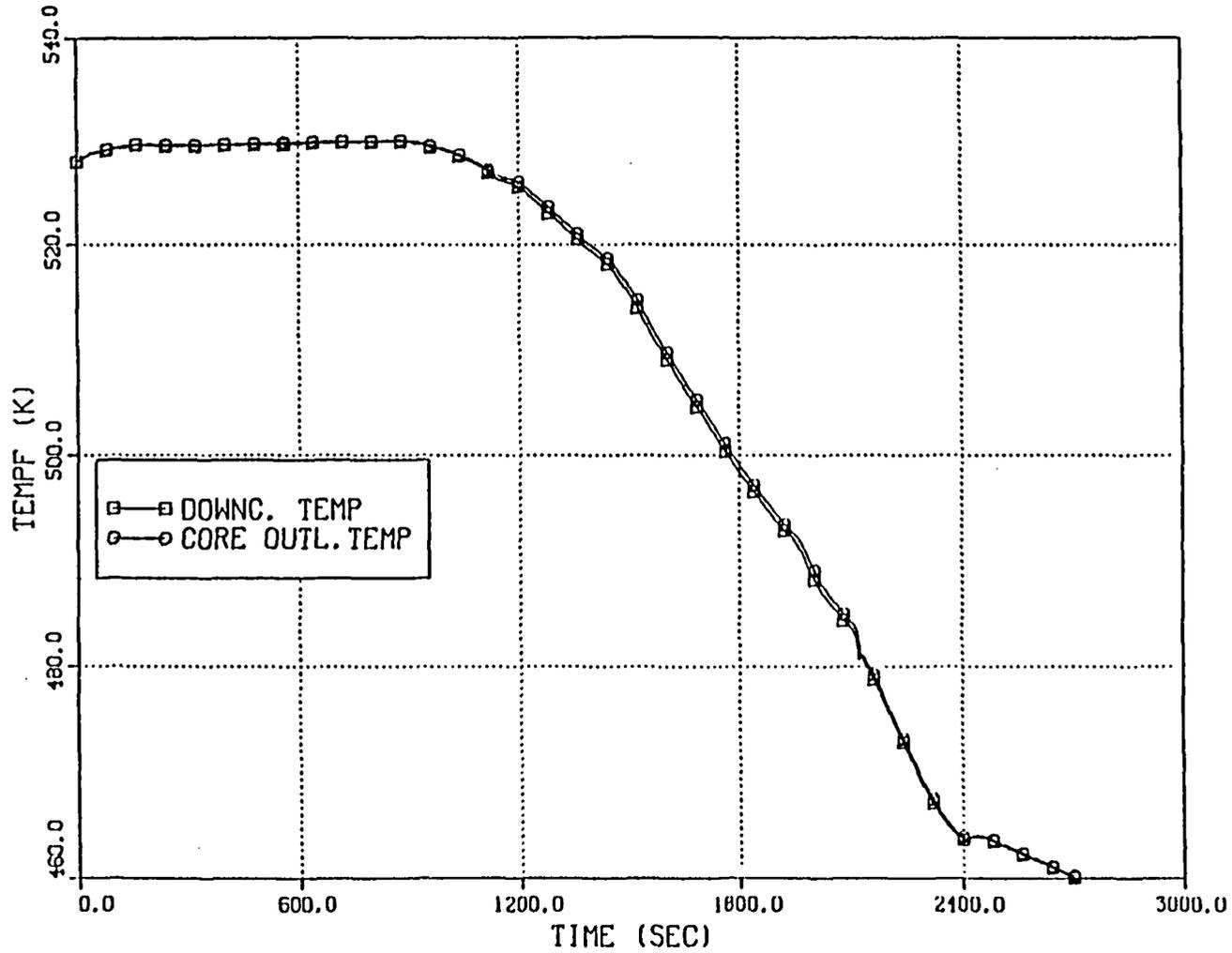
5. FIG 4.1 PRESSURE EVOLUTION IN HOT LEG A



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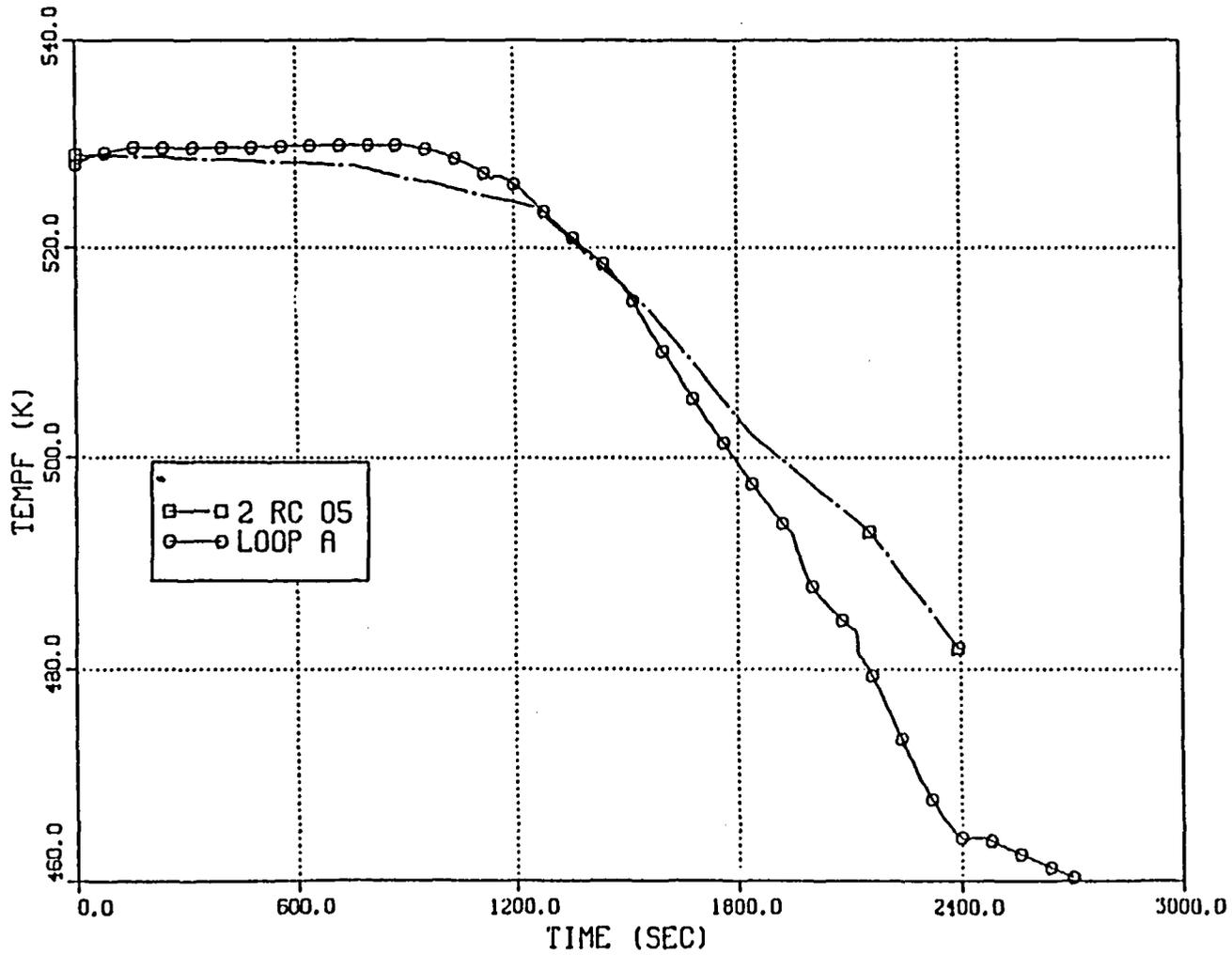
5. FIG 5.2 TEMPERATURE EVOLUTION IN REACTOR VESSEL



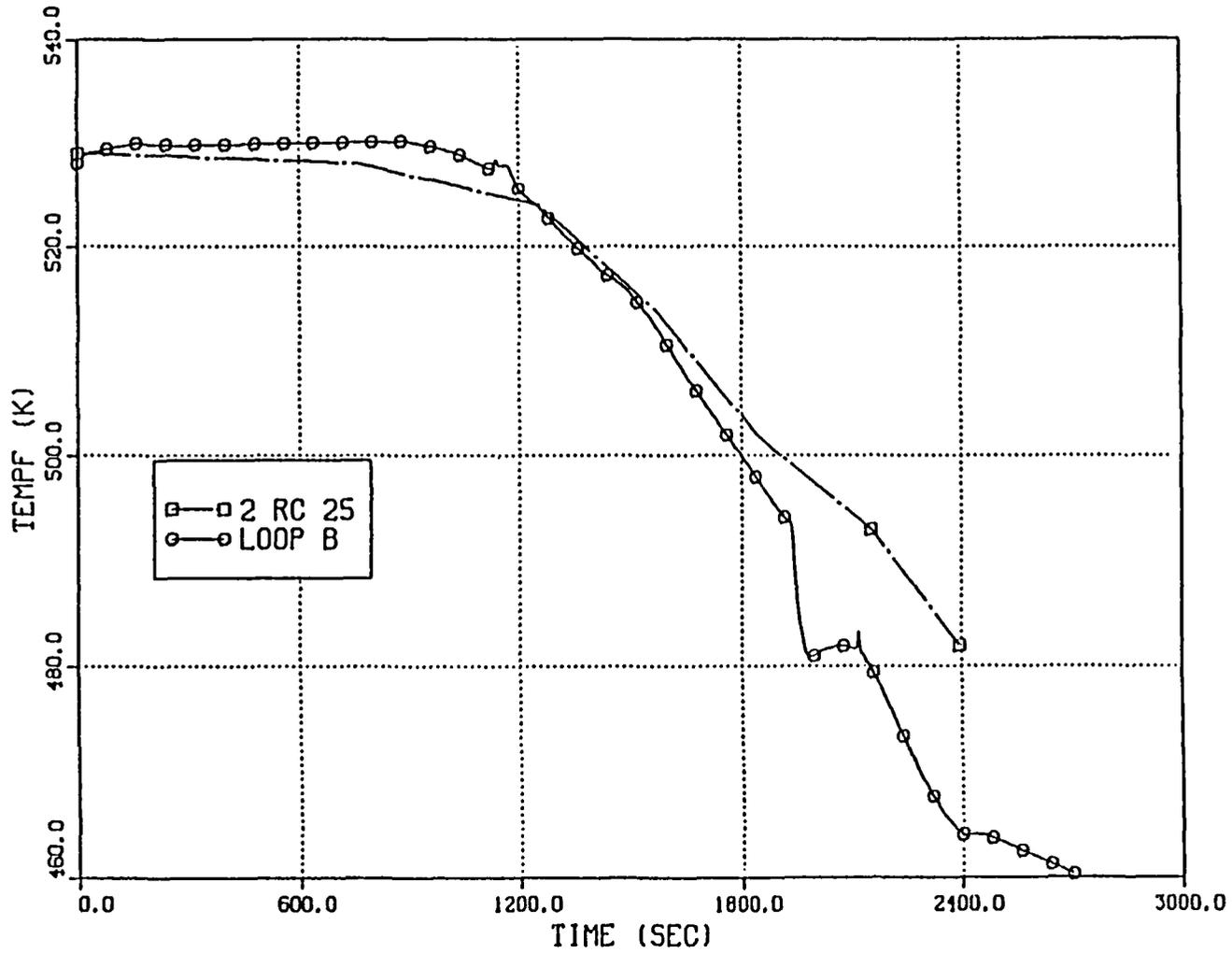
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DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28..

5. FIG 5.4 TEMPERATURE EVOLUTION IN HOT LEG A



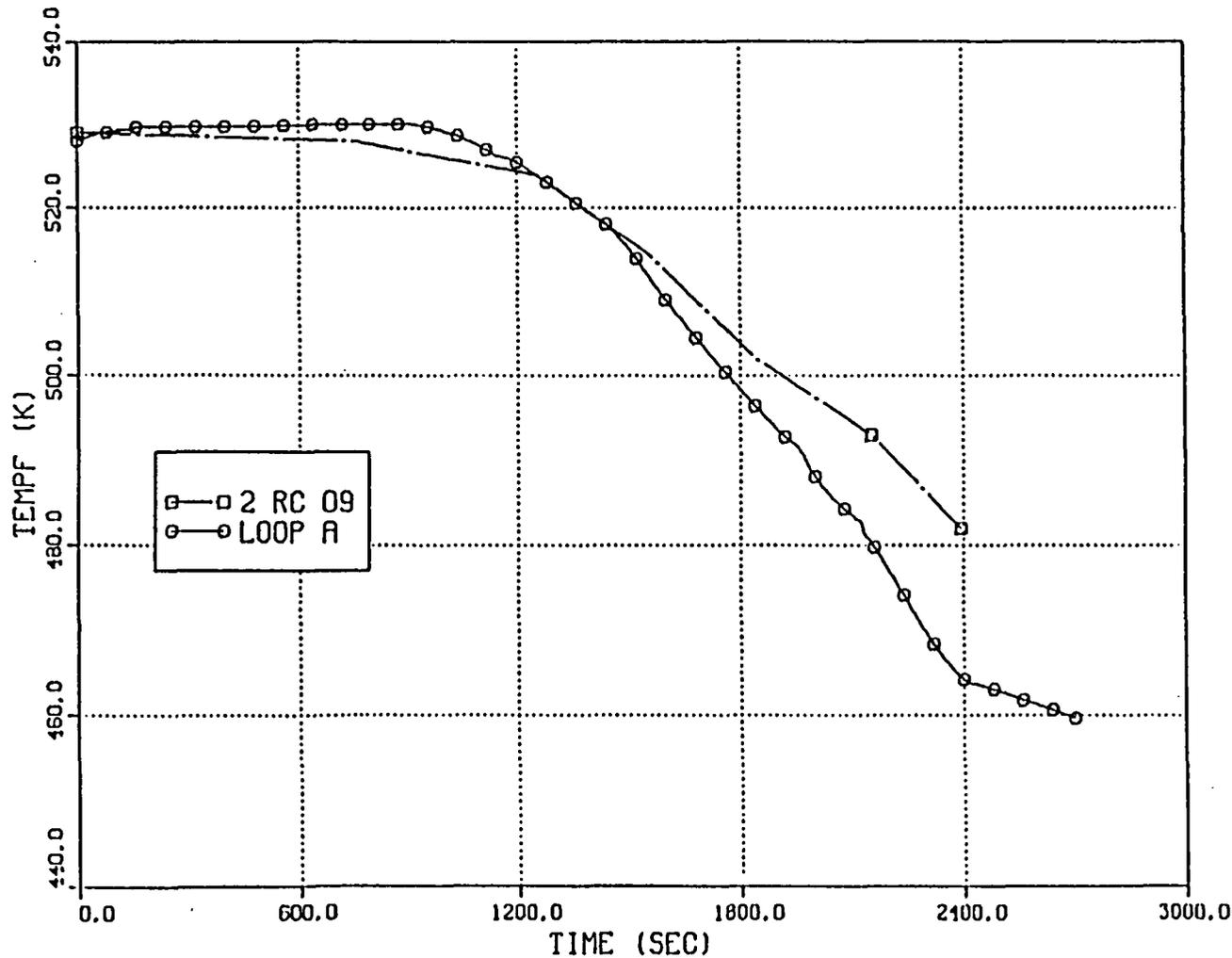
RELAP5/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM
DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.
5. FIG 5.5 TEMPERATURE EVOLUTION IN HOT LEG B



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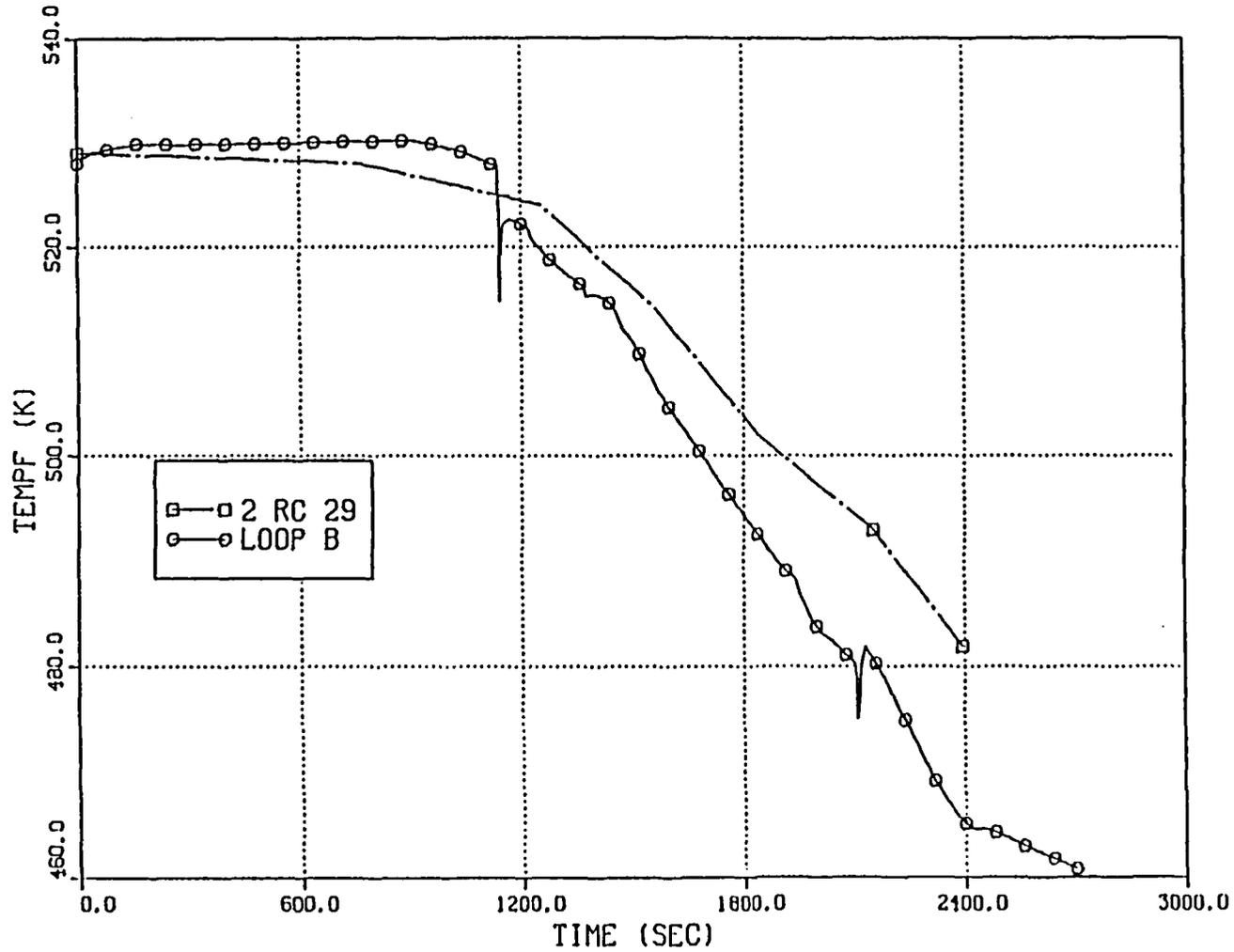
5. FIG 5.6 TEMPERATURE EVOLUTION IN COLD LEG A



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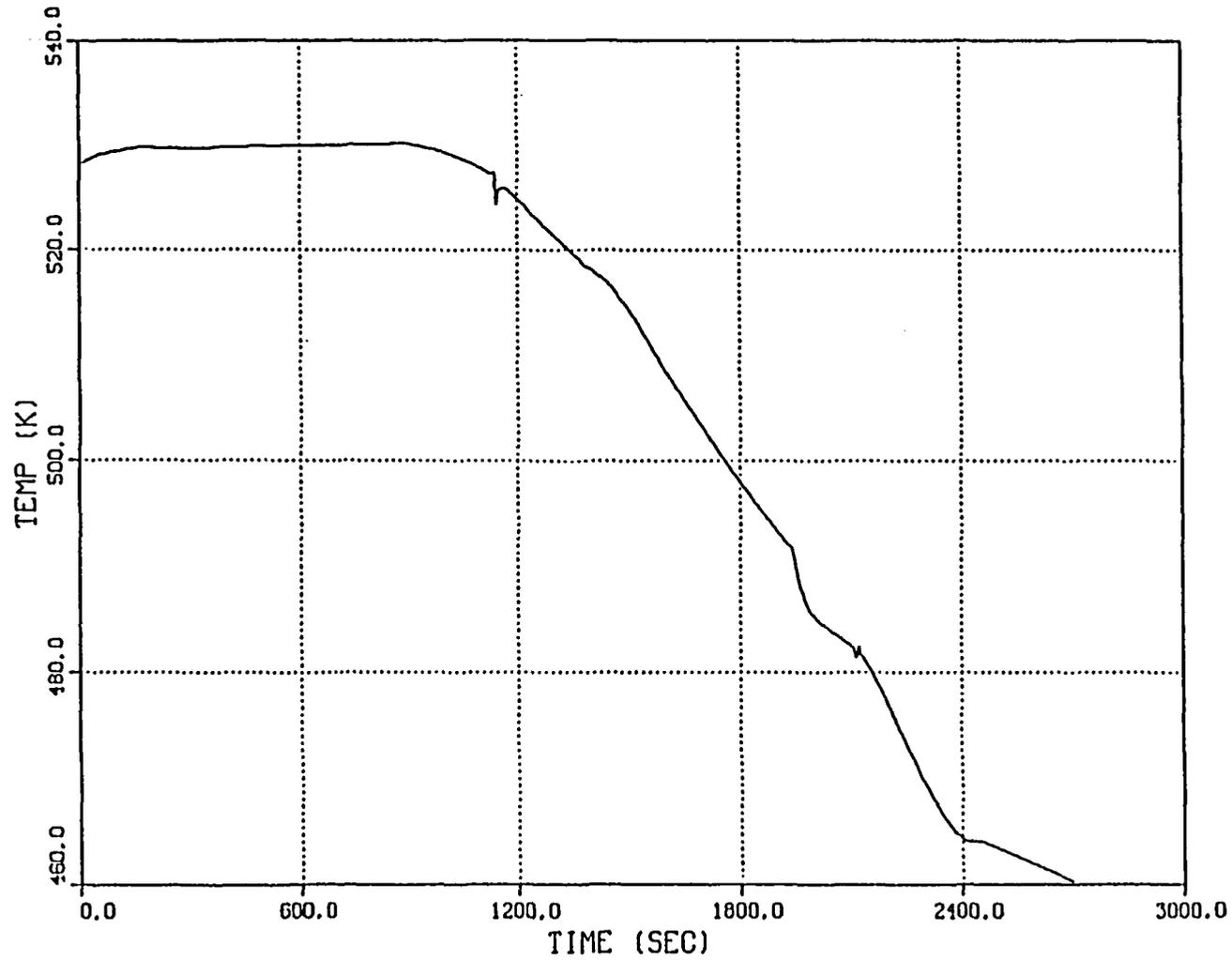
5. FIG 5.7 TEMPERATURE EVOLUTION IN COLD LEG B



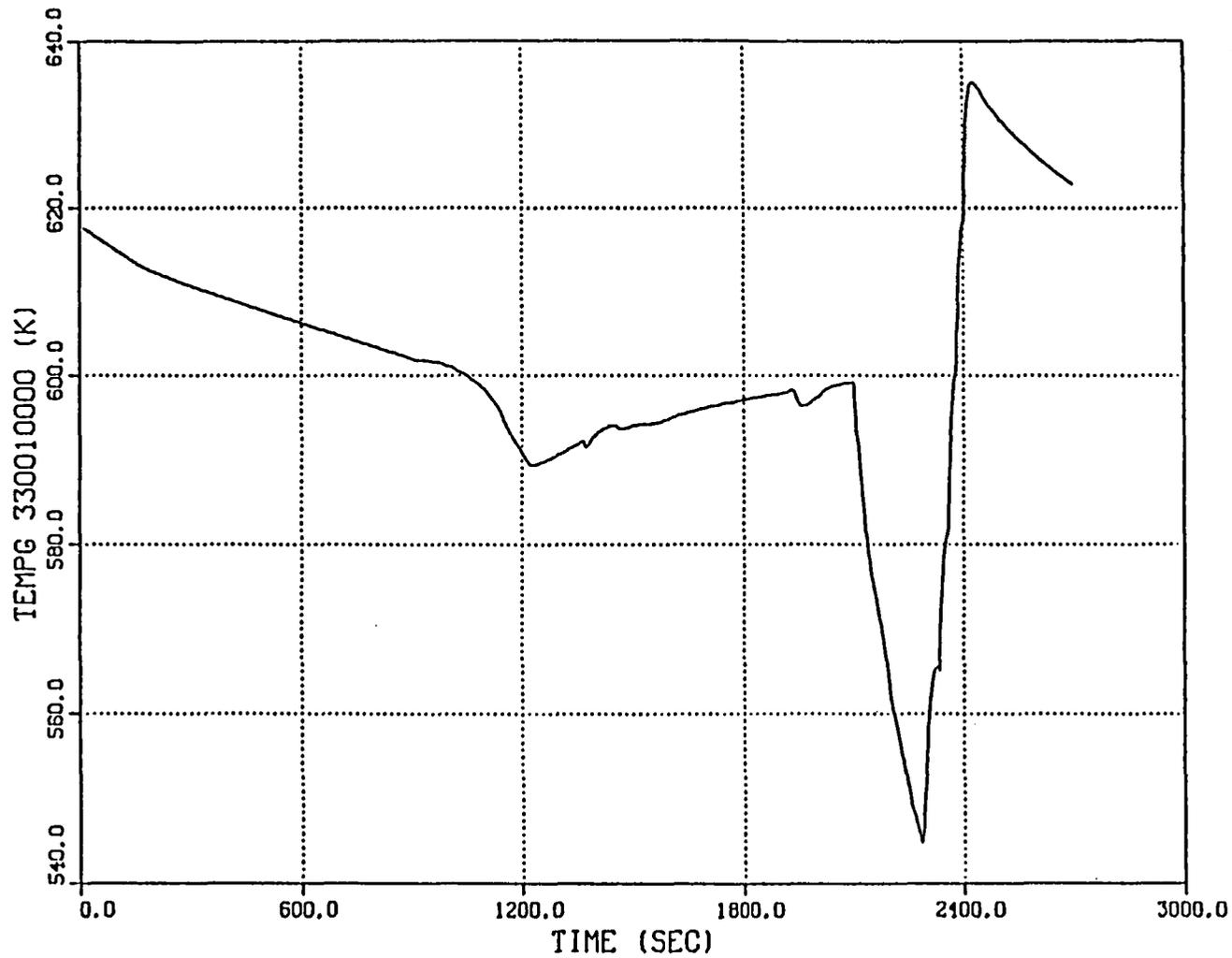
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5. FIG 5.8 AVERAGED PRIMARY TEMPERATURE EVOLUTION



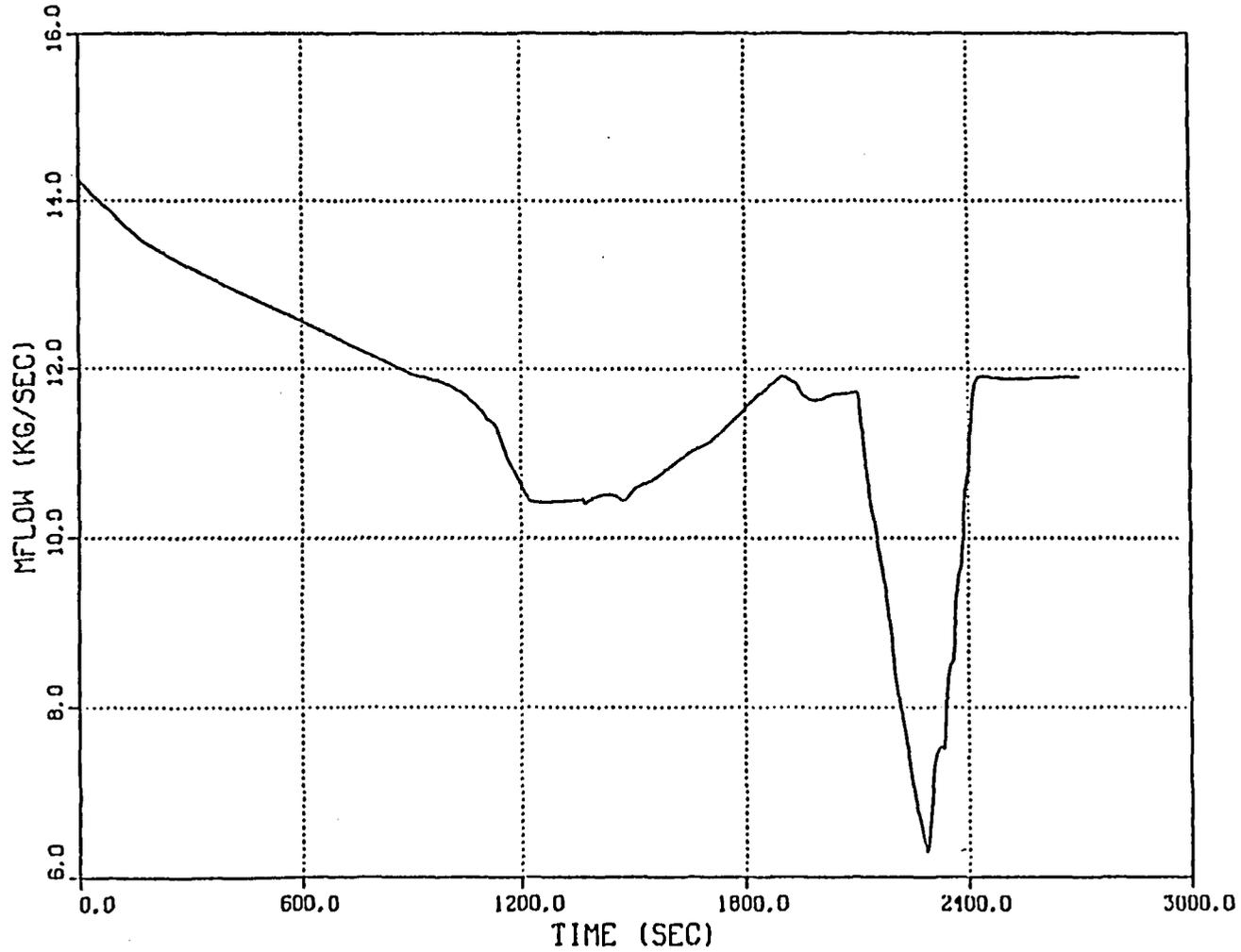
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DOEL2 SGTR INCIDENT 25 JUNE 1979 05/09/28..
5. FIG 5.9 PRESSURIZER STEAM TEMPERATURE EVOLUTION



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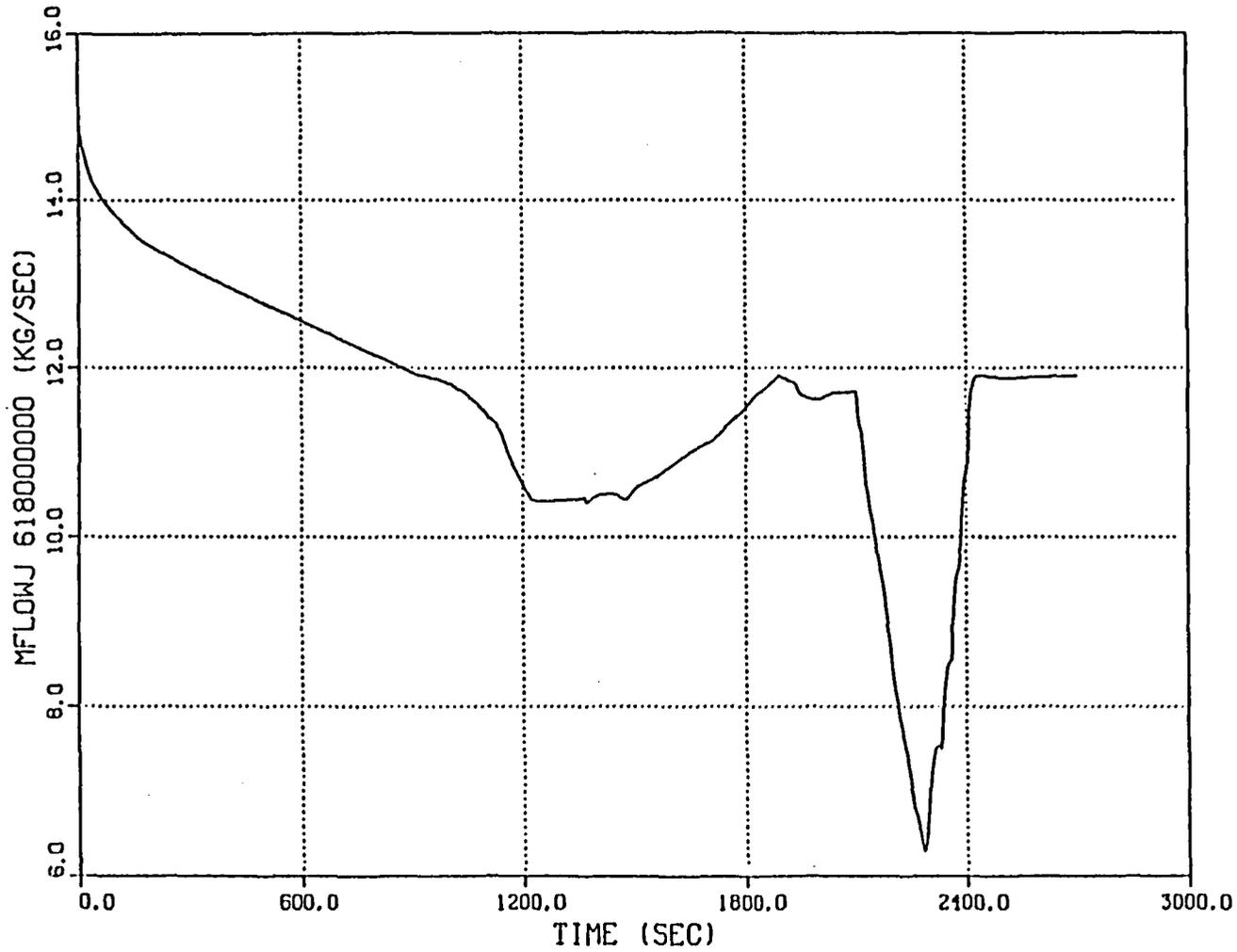
5. FIG 6.1 BREAK MASS FLOW RATE (CORRECTED)



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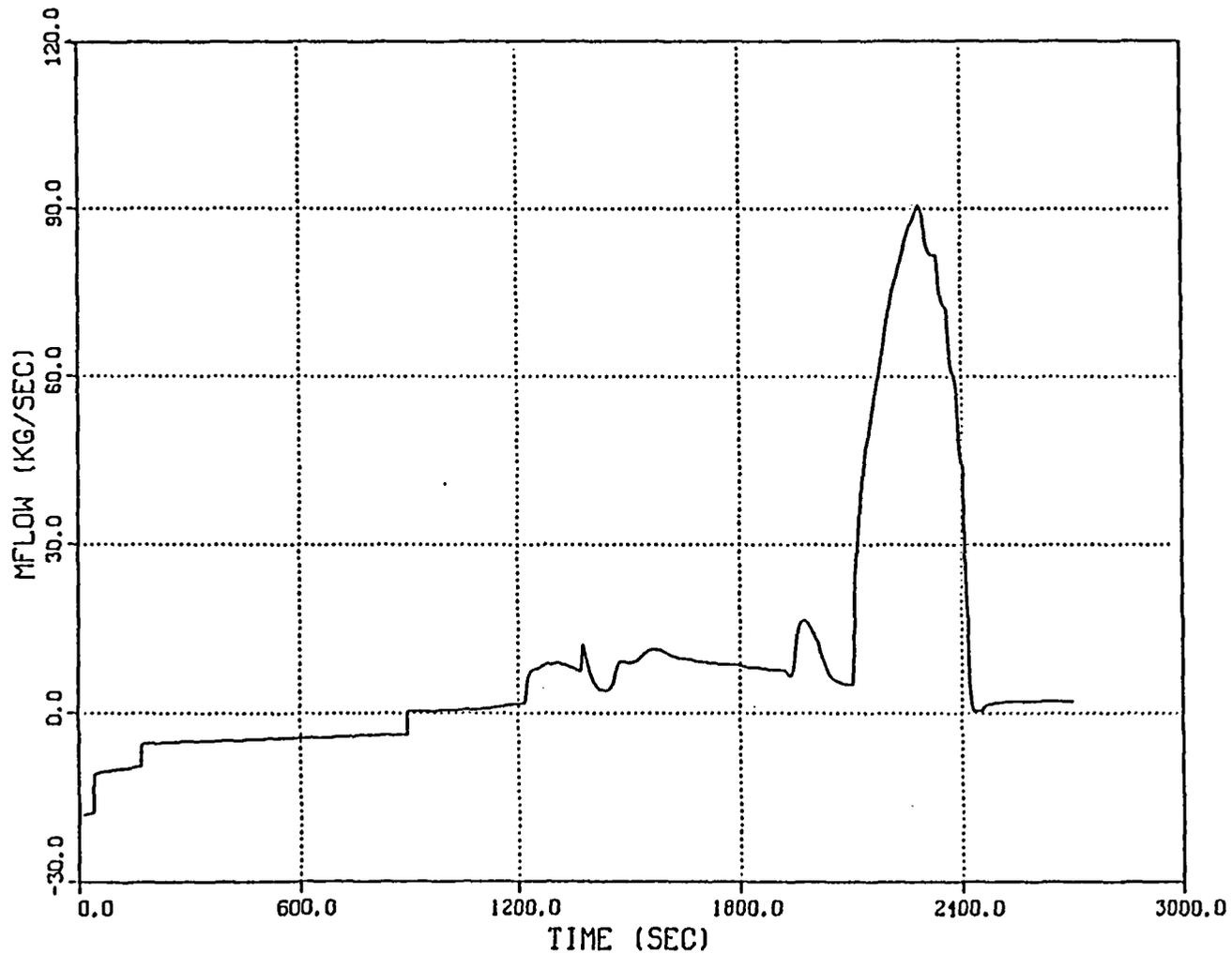
5. FIG 6.2 BREAK MASS FLOW RATE



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DOEL2 S6TR INCIDENT 25 JUNE 1979 85/09/28.

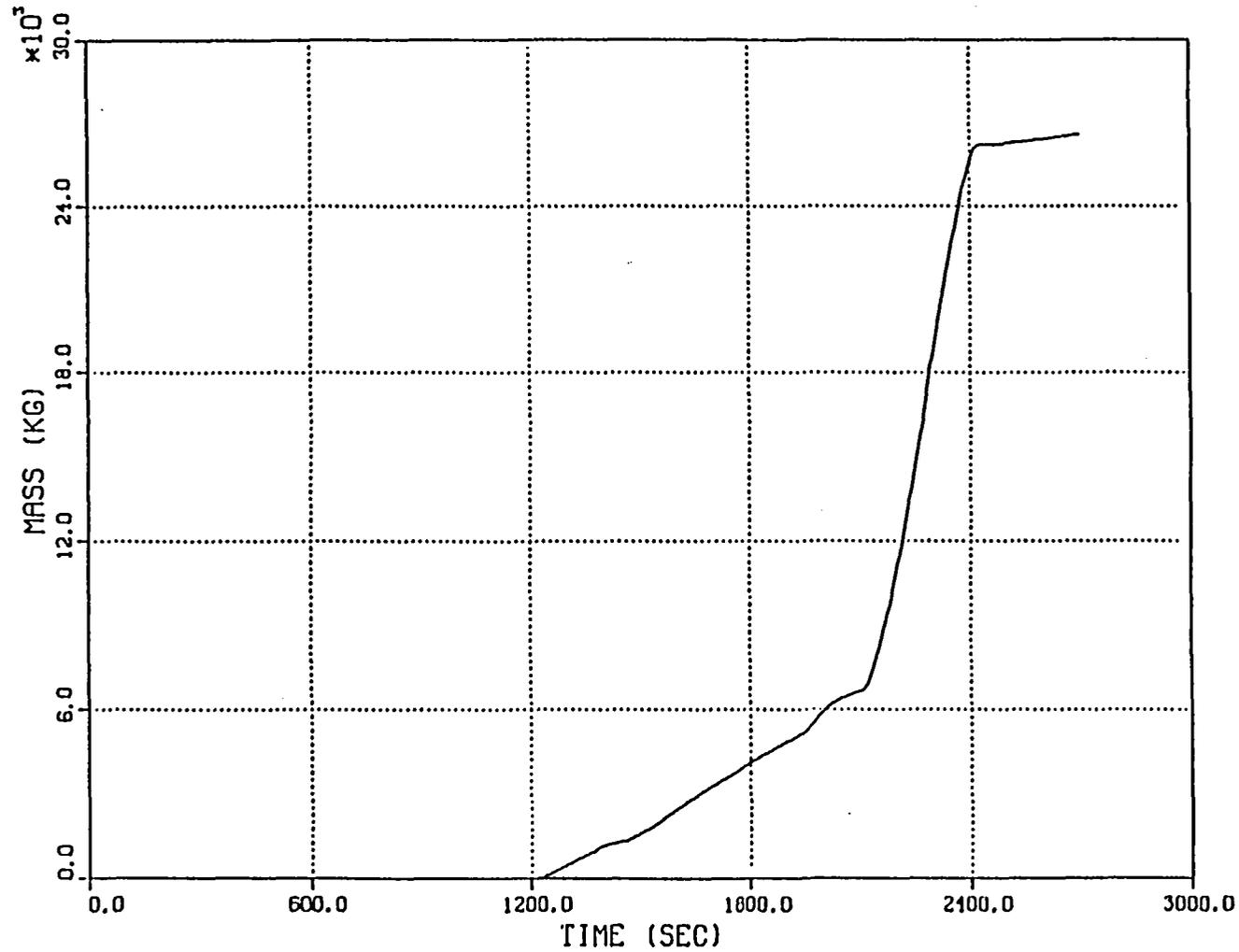
5. FIG 6.3 PRIMARY SYSTEM MASS ADDITION AND DEPLETION RATES



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DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

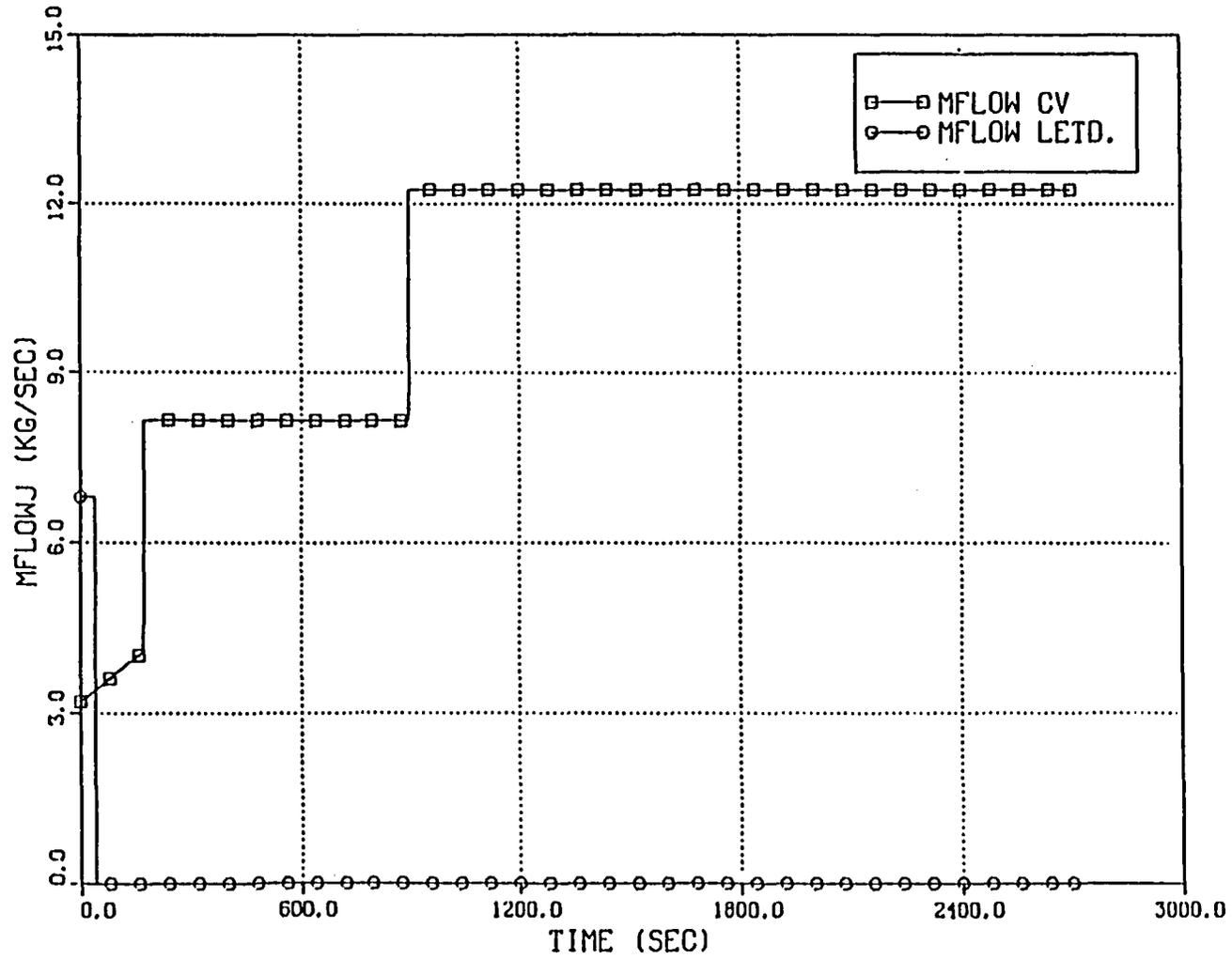
5. FIG 6.4 INTEGRAL OF HPSI MASS FLOW RATE



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DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

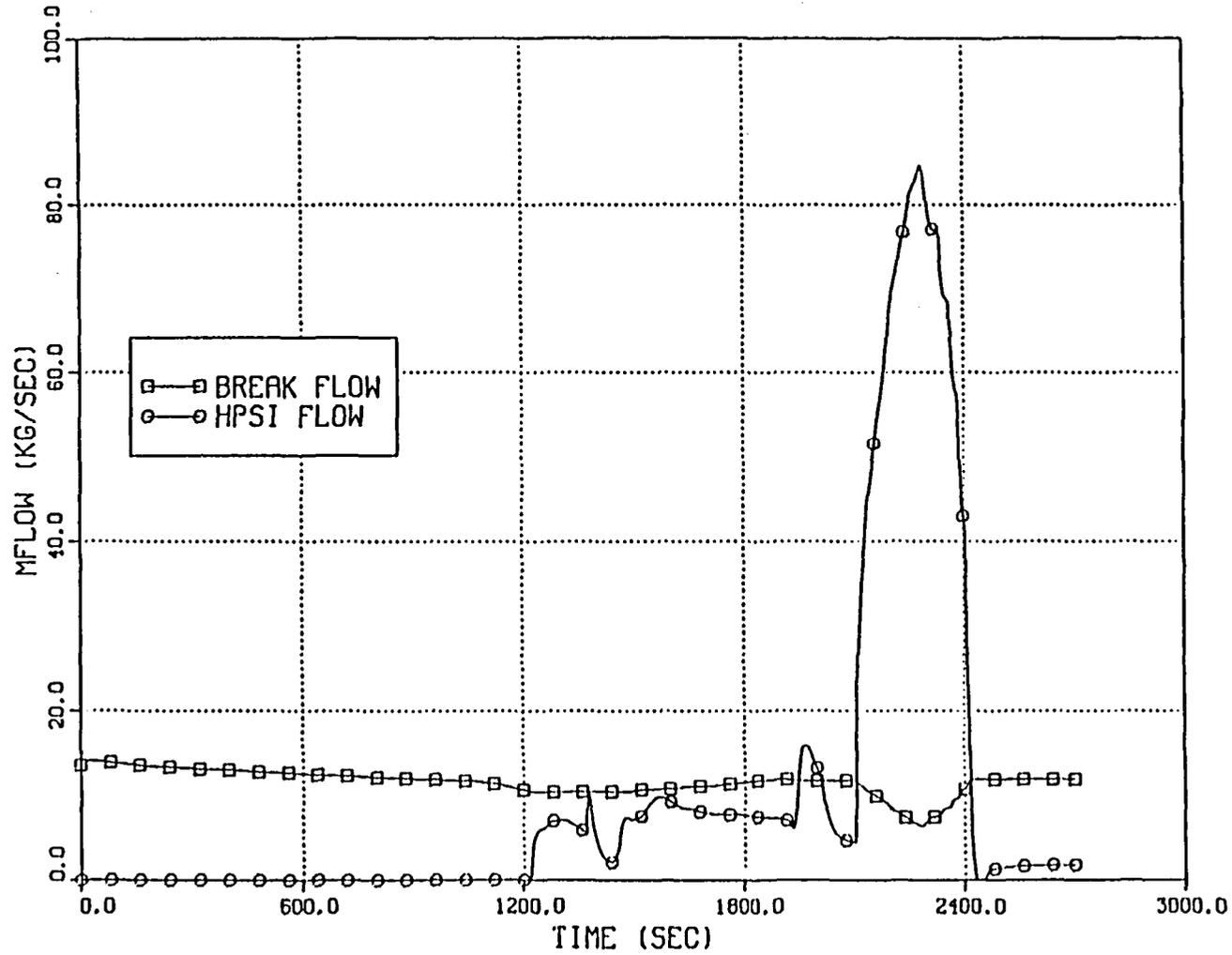
5. FIG 6.5 CHARGING AND LETDOWN MASS FLOW RATE



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DOEL2 SGTR INCIDENT 25 JUNE 1979 05/09/28.

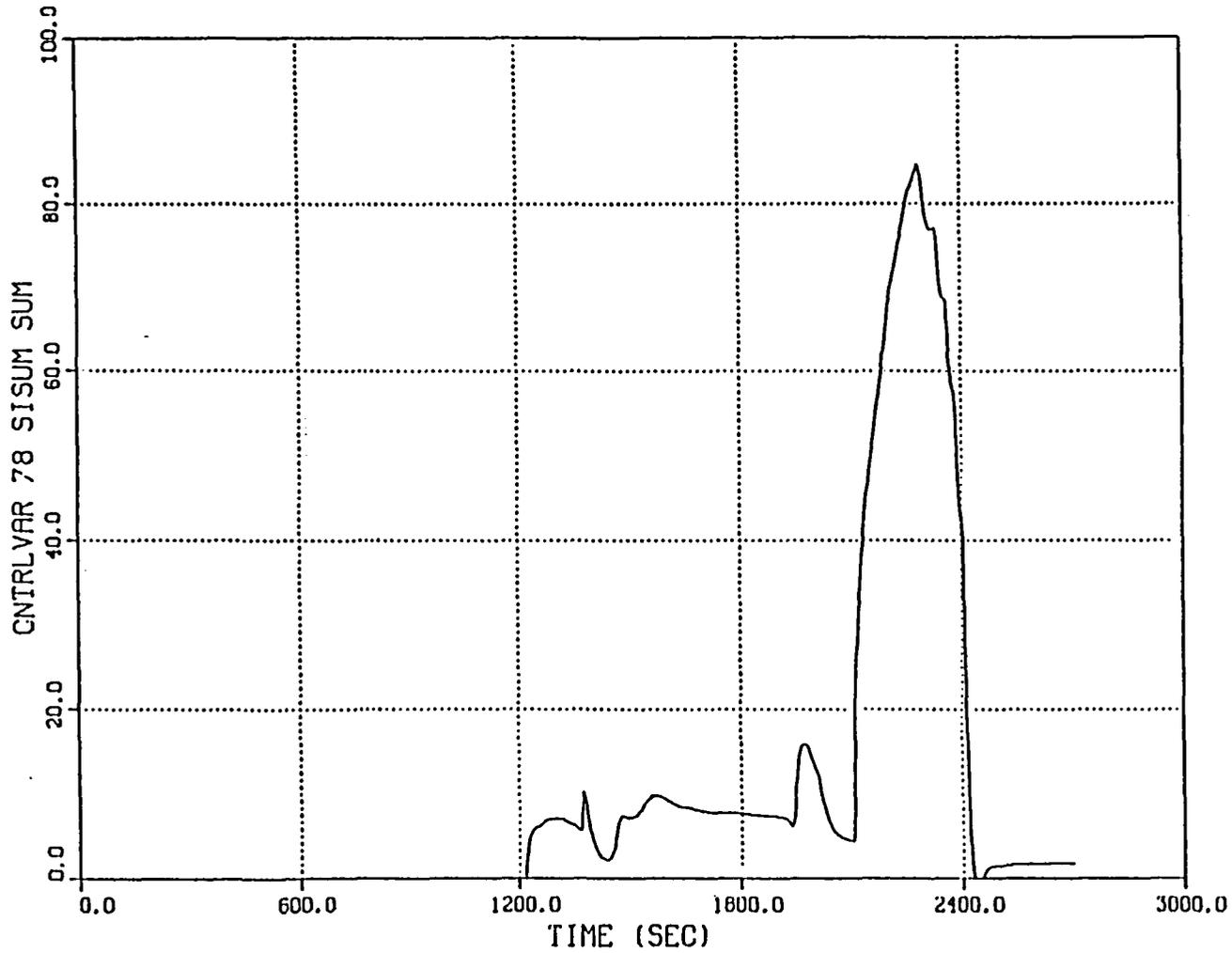
5. FIG 6.6 BREAK AND HPSI MASS FLOW RATE



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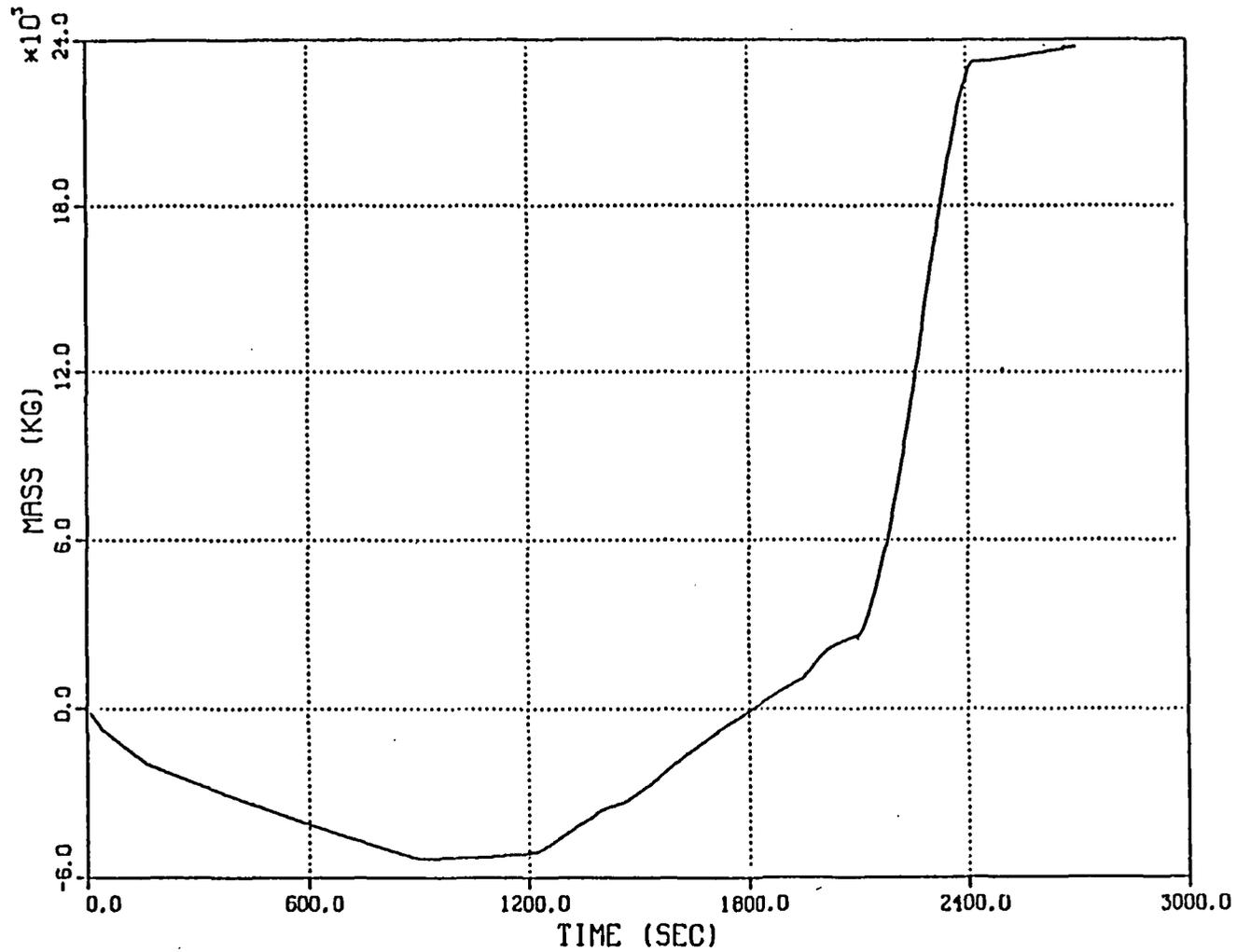
5. FIG 6.7 HPSI MASS FLOW RATE



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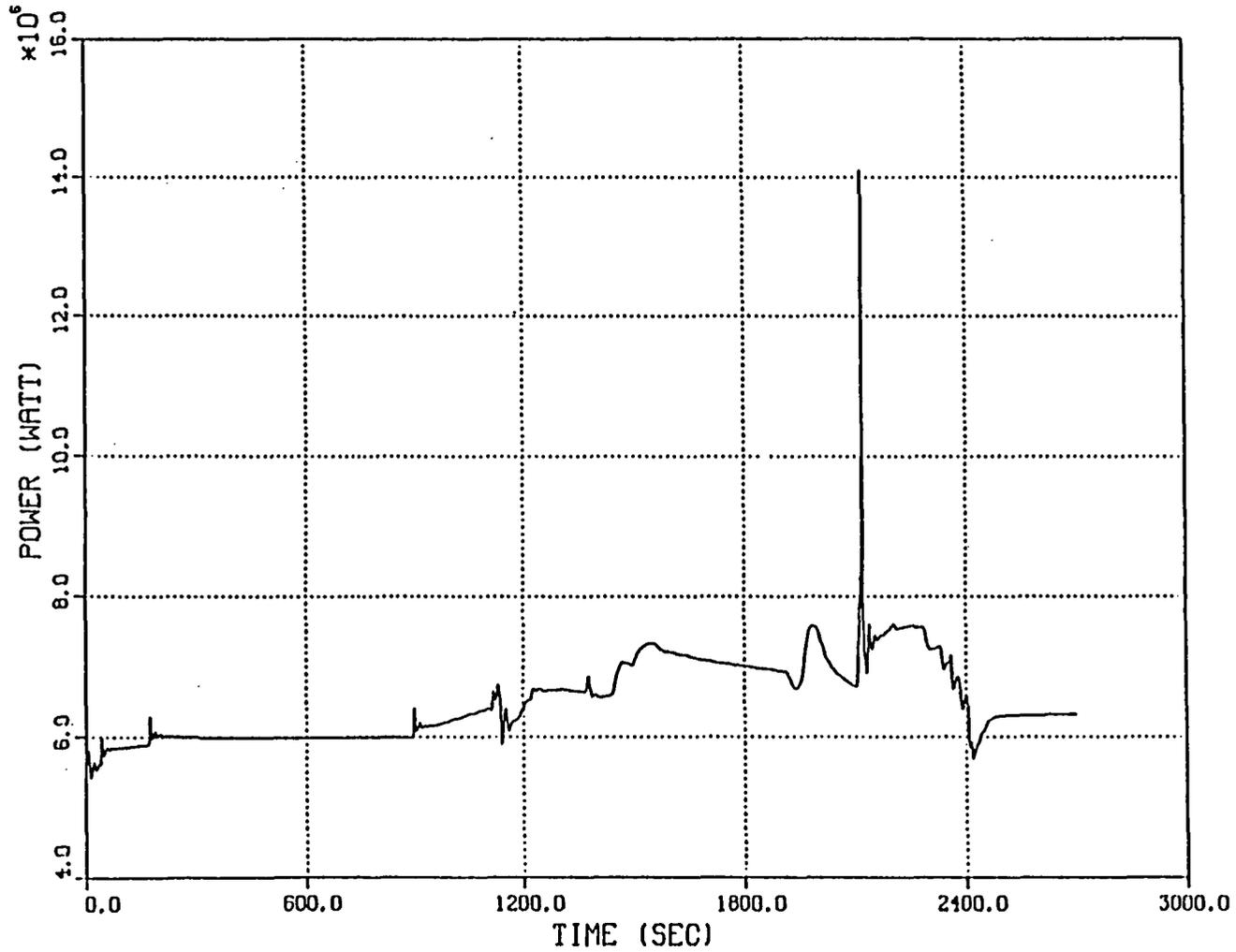
5. FIG 6.10 MASS BALANCE IN PRIMARY SYSTEM



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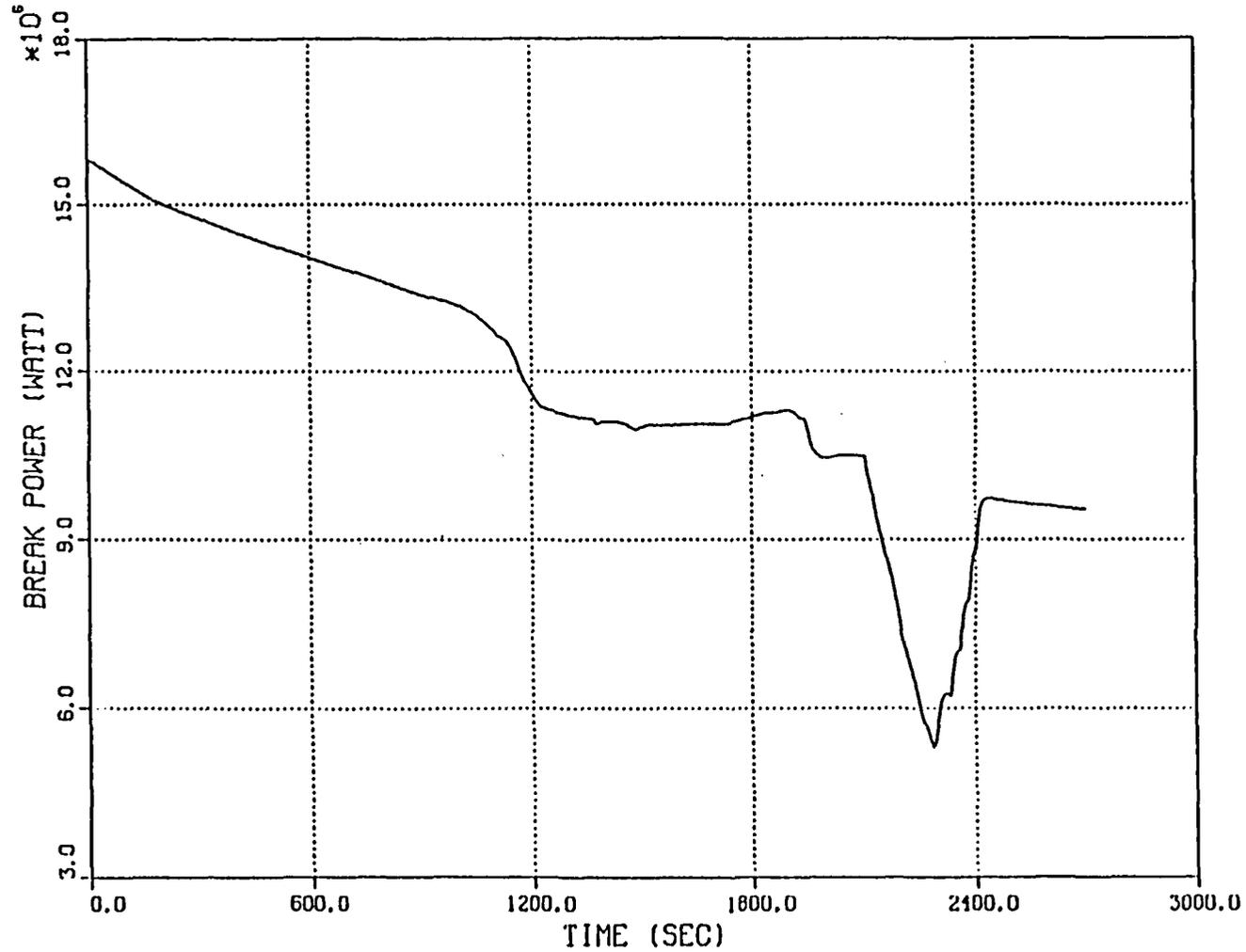
5. FIG 7.1 CORE THERMAL POWER



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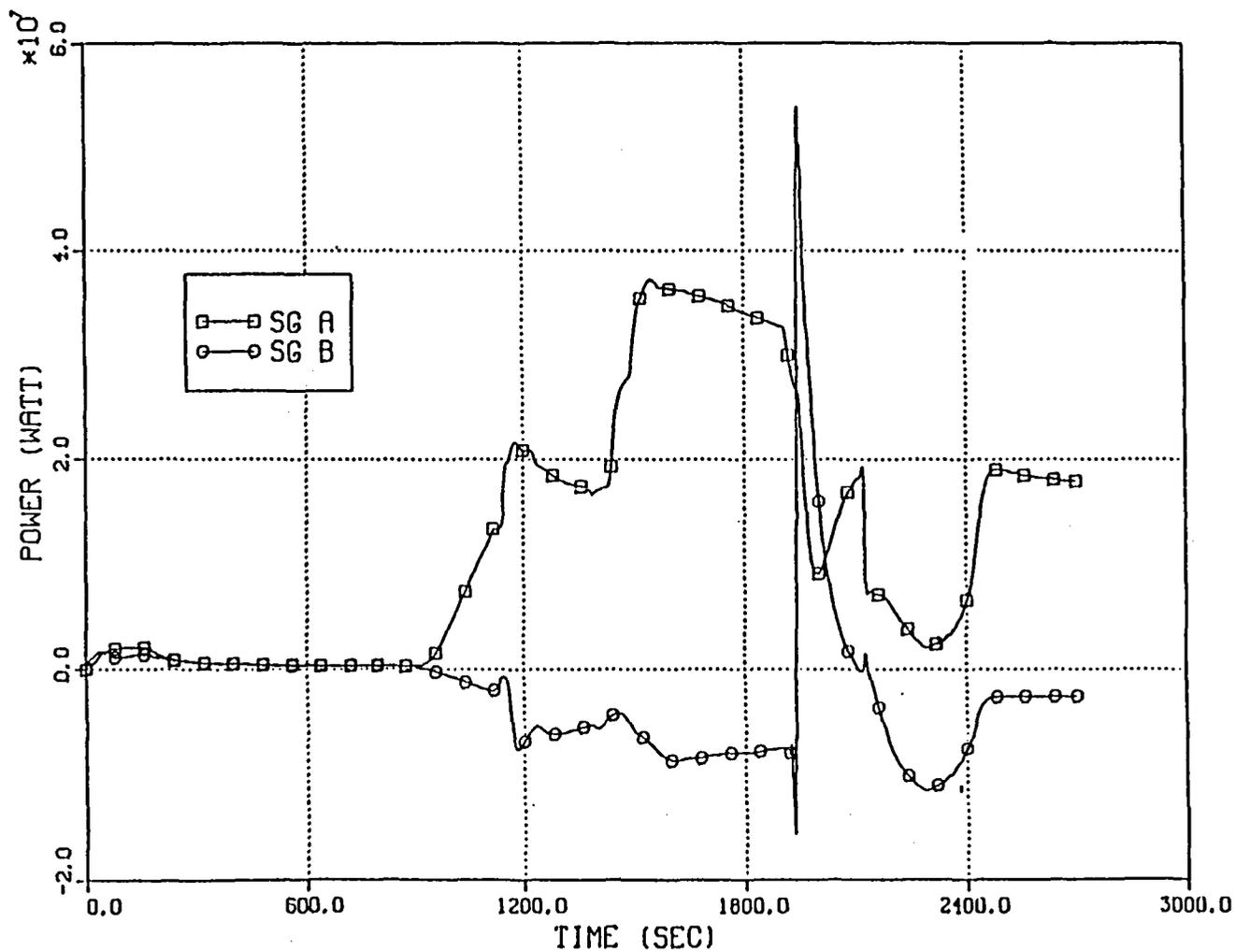
5. FIG 7.2 BREAK ENERGY EVOLUTION



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DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

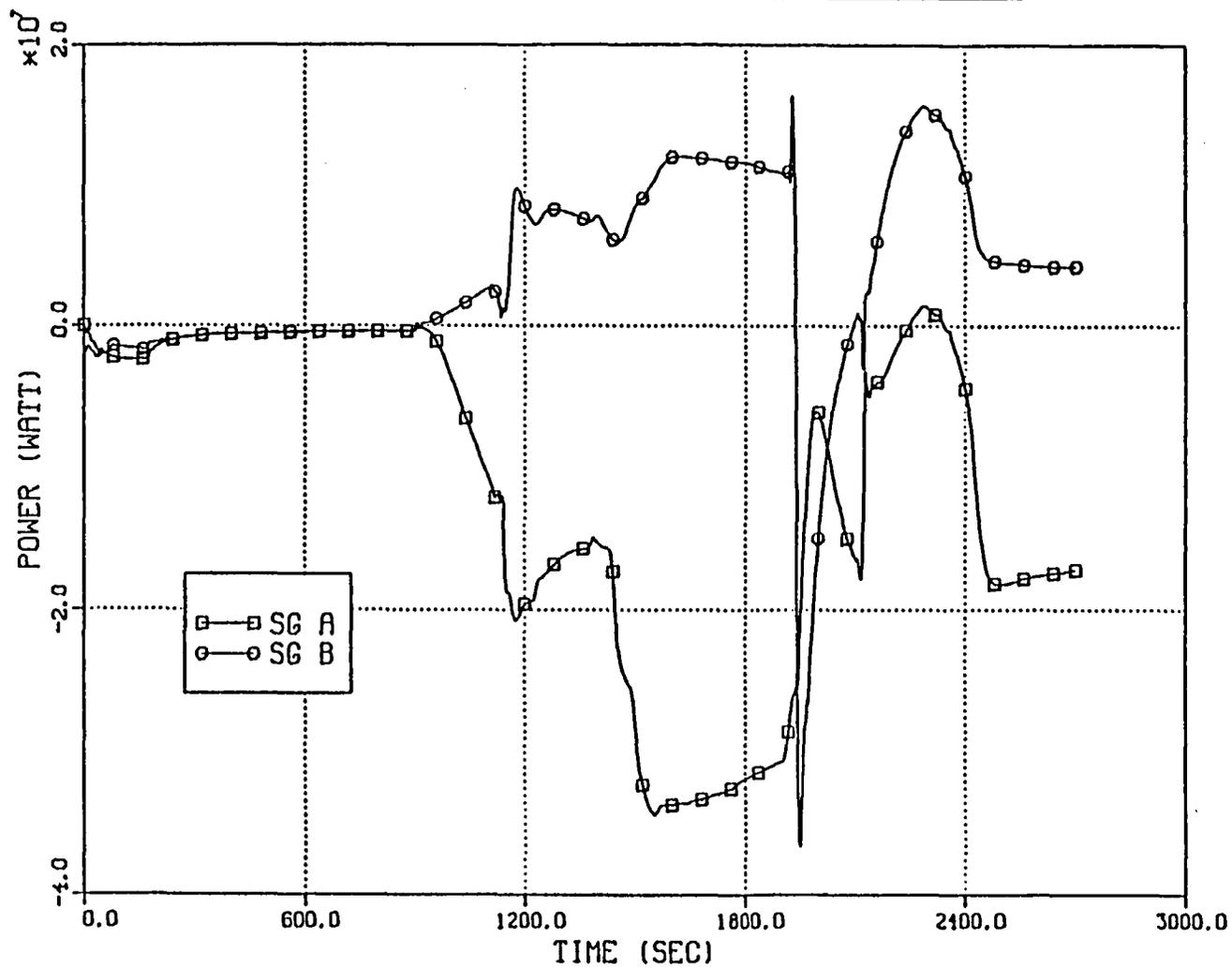
5. FIG 7.3 HEAT TRANSFER RATE FROM PRIMARY TO SG



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DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

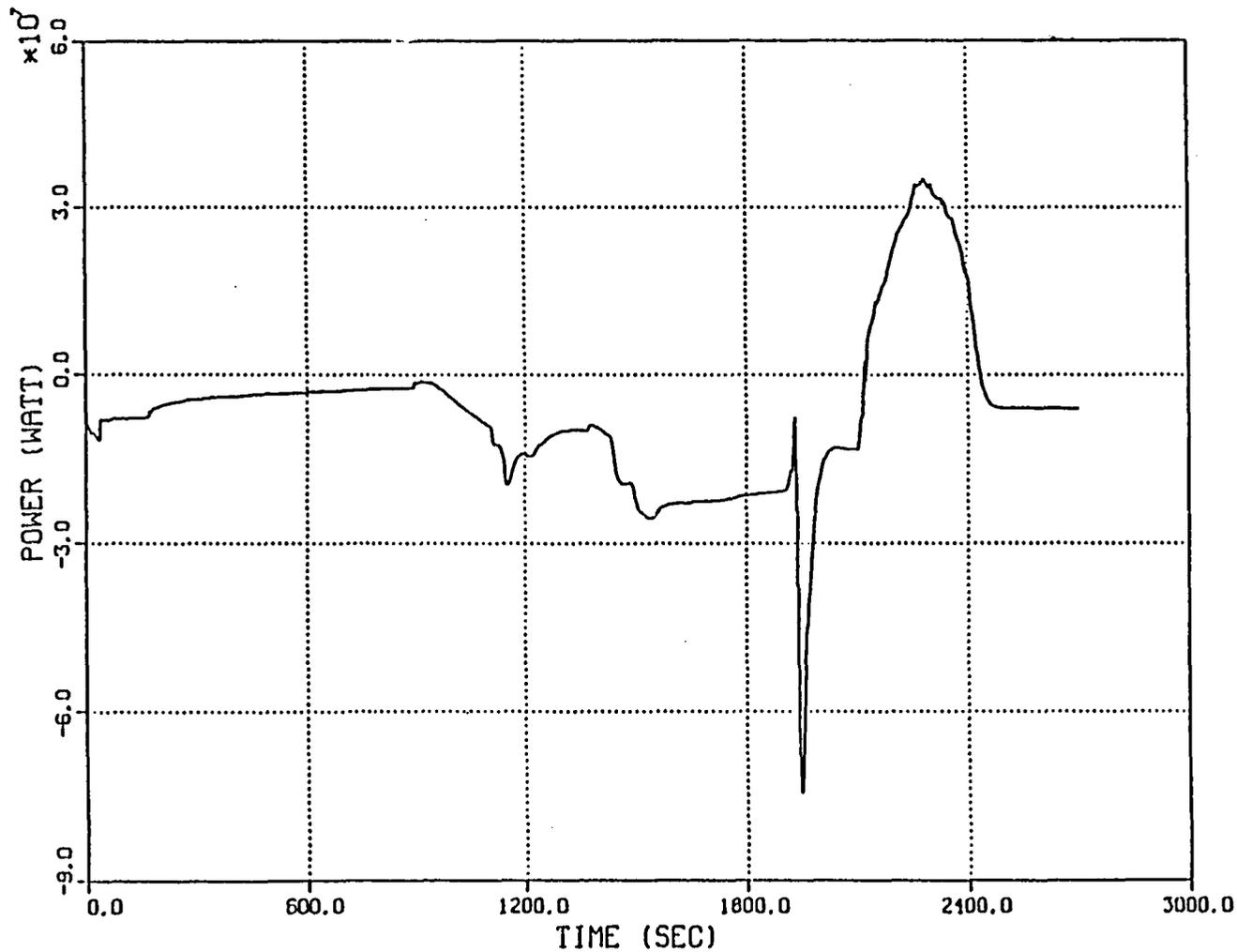
5. FIG 7.4 HEAT TRANSFER RATE FROM PRIMARY TO SG'S



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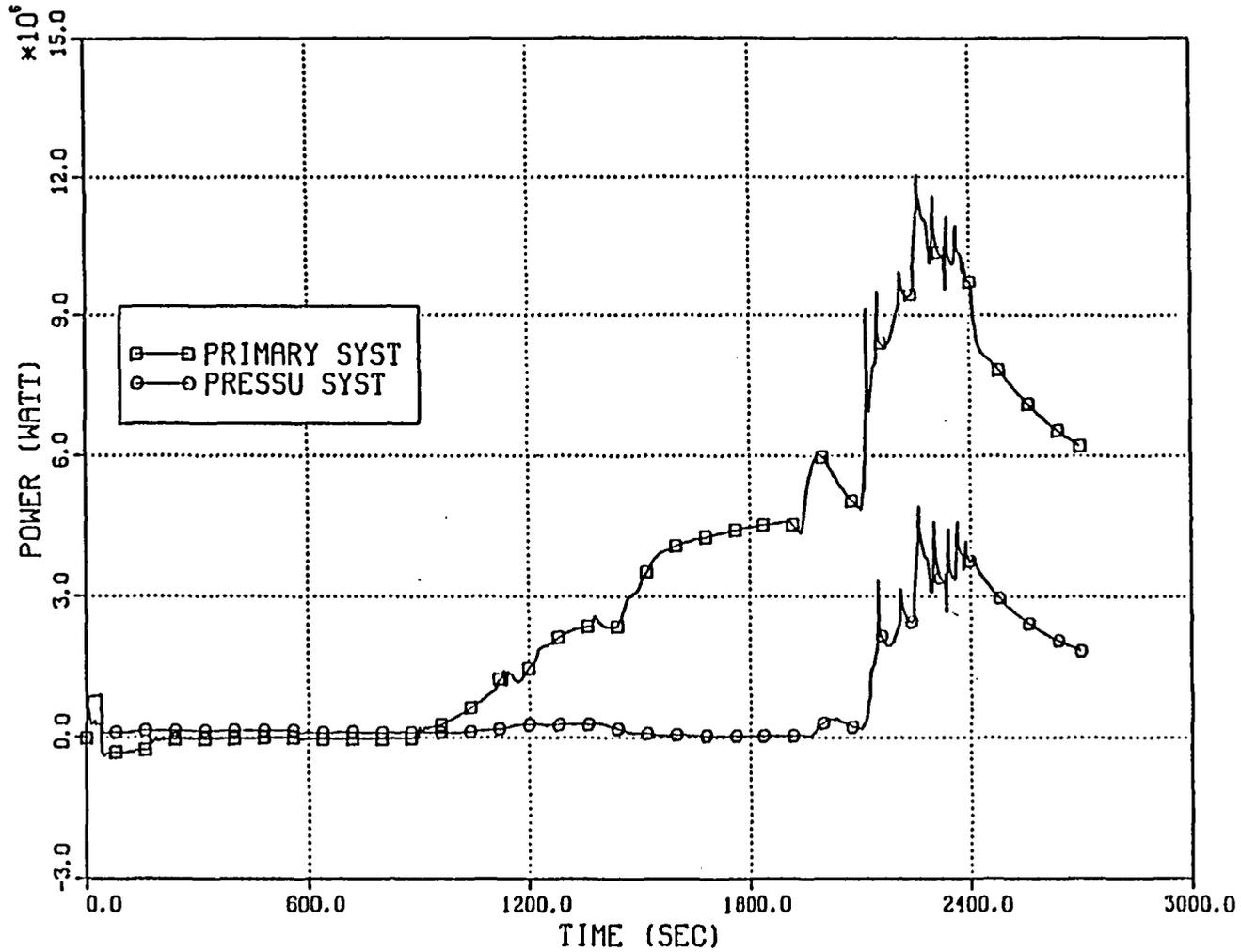
DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

5. FIG 7.5 PRIMARY SYSTEM ENERGY BALANCE



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DOEL2 SGR INCIDENT 25 JUNE 1979 85/09/28.

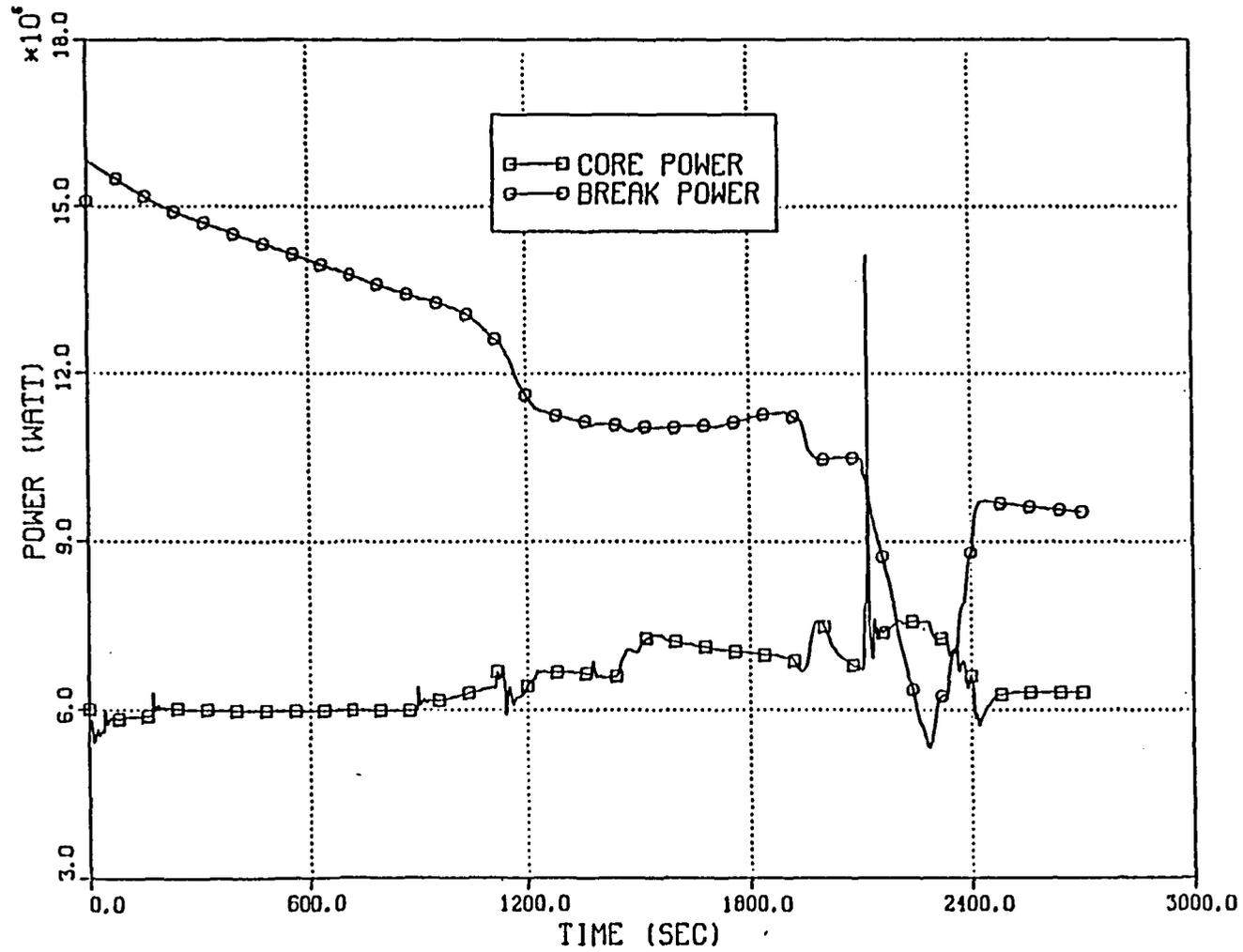
5. FIG 7.6 HEAT ADDITION RATE FROM PRIMARY METAL STRUCTURES



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DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

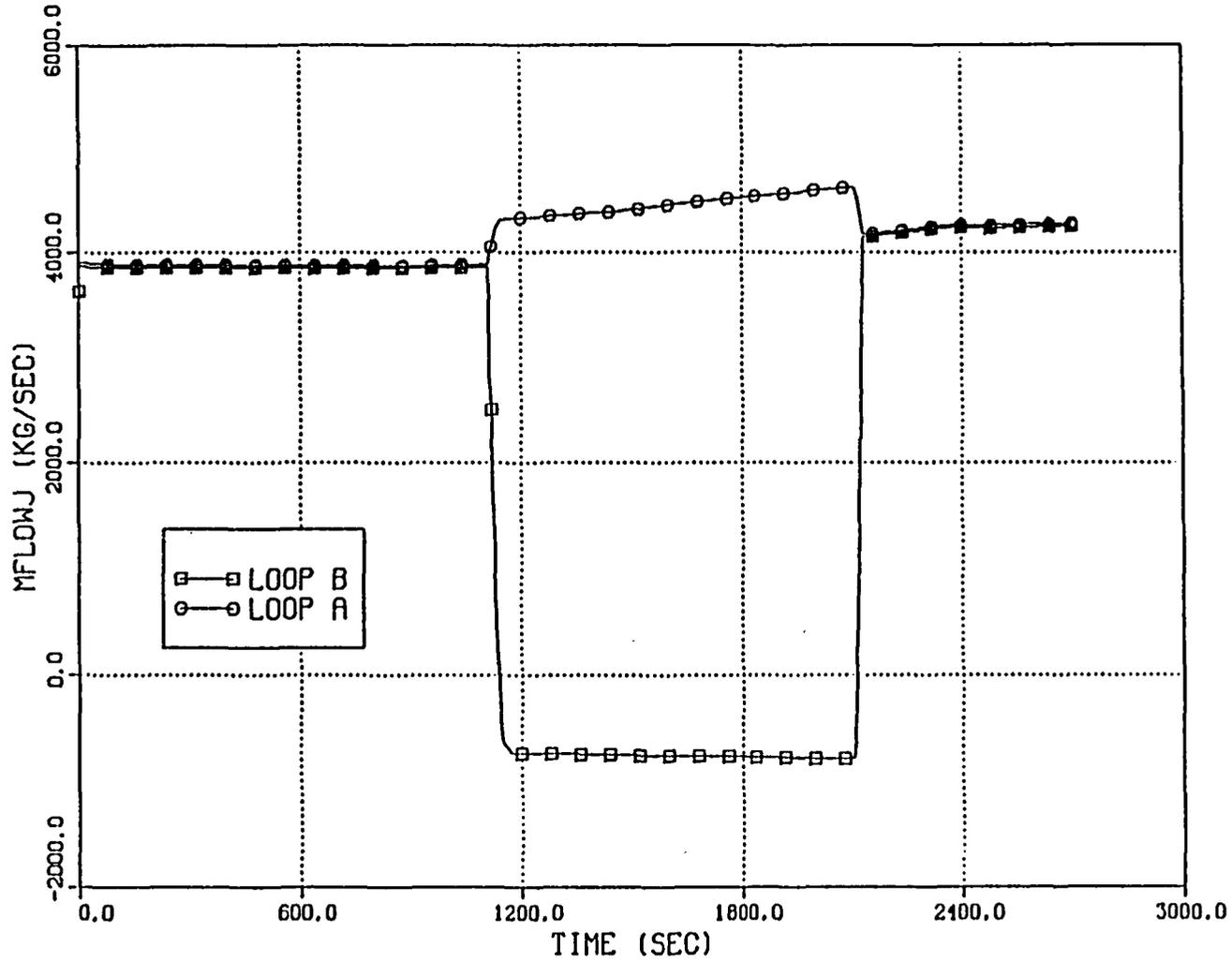
5. FIG 7.7 THERMAL CORE AND BREAK POWER



RELAP5/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM

DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

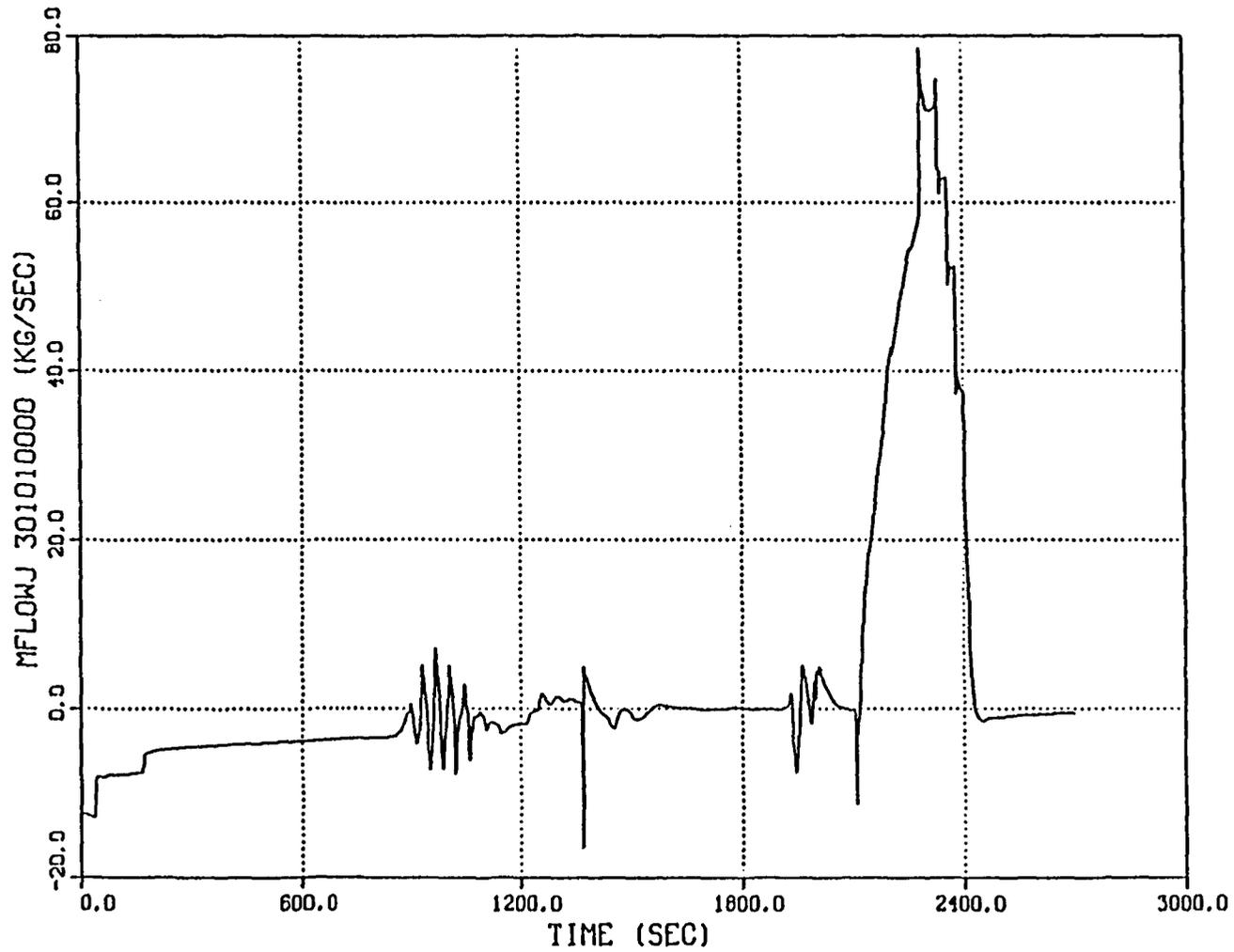
5. FIG 8.1 COLD LEG MASS FLOW RATE (VESSEL INLET)



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DOEL2 SGTR INCIDENT 25 JUNE 1979 05/09/28.

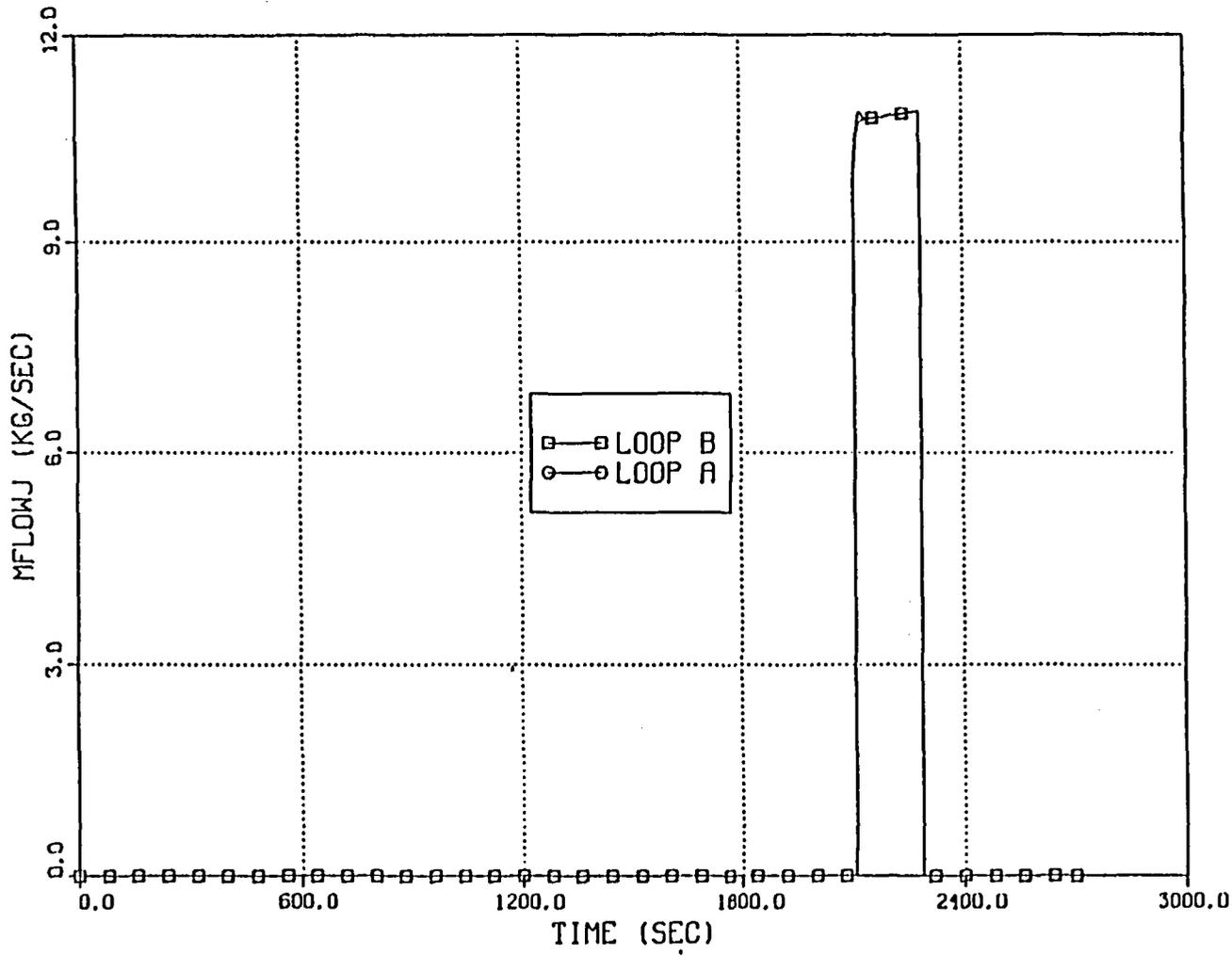
5. FIG 8.3 SURGELINE MASS FLOW RATE



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DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28..

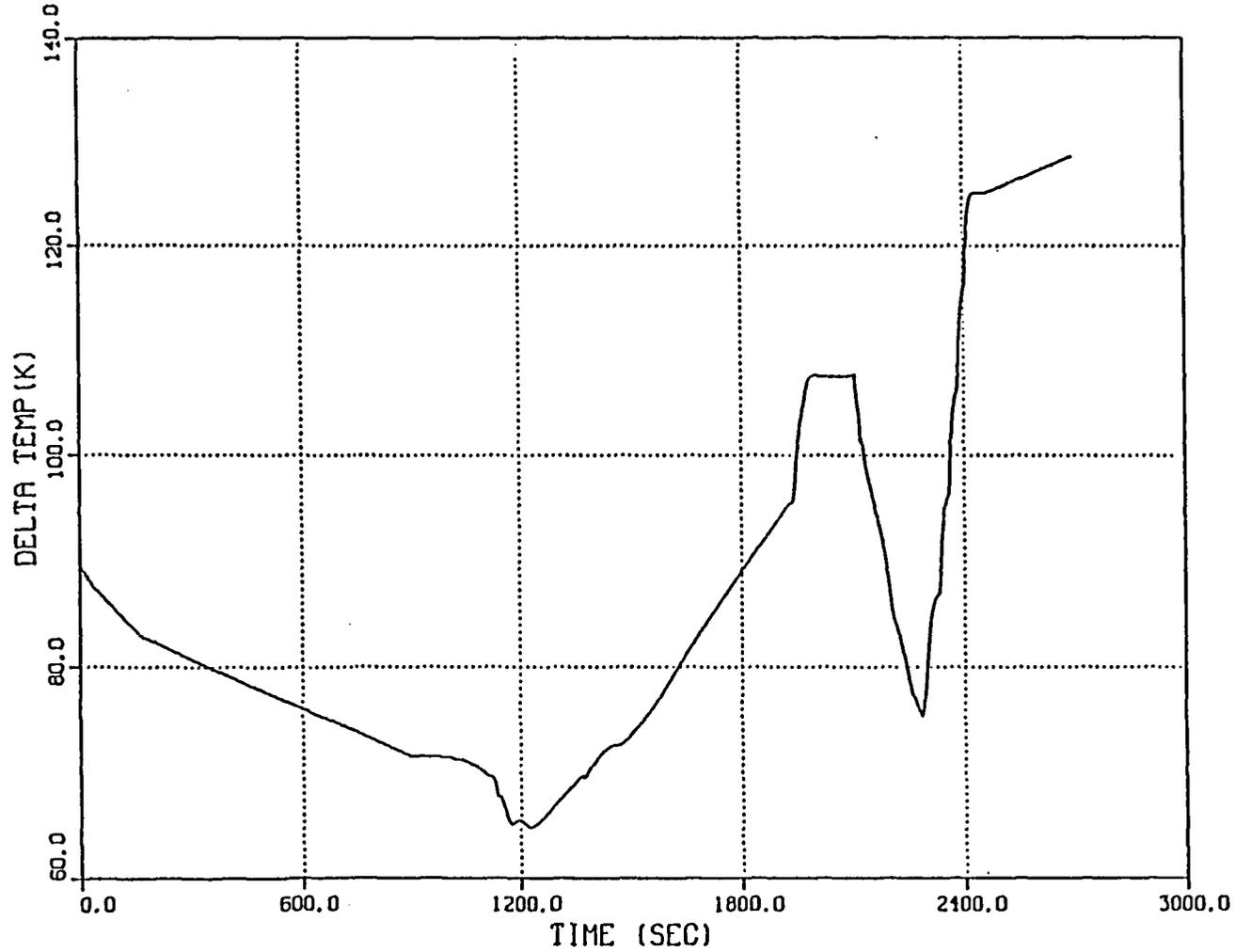
5. FIG 8.5 SPRAY MASS FLOW RATE



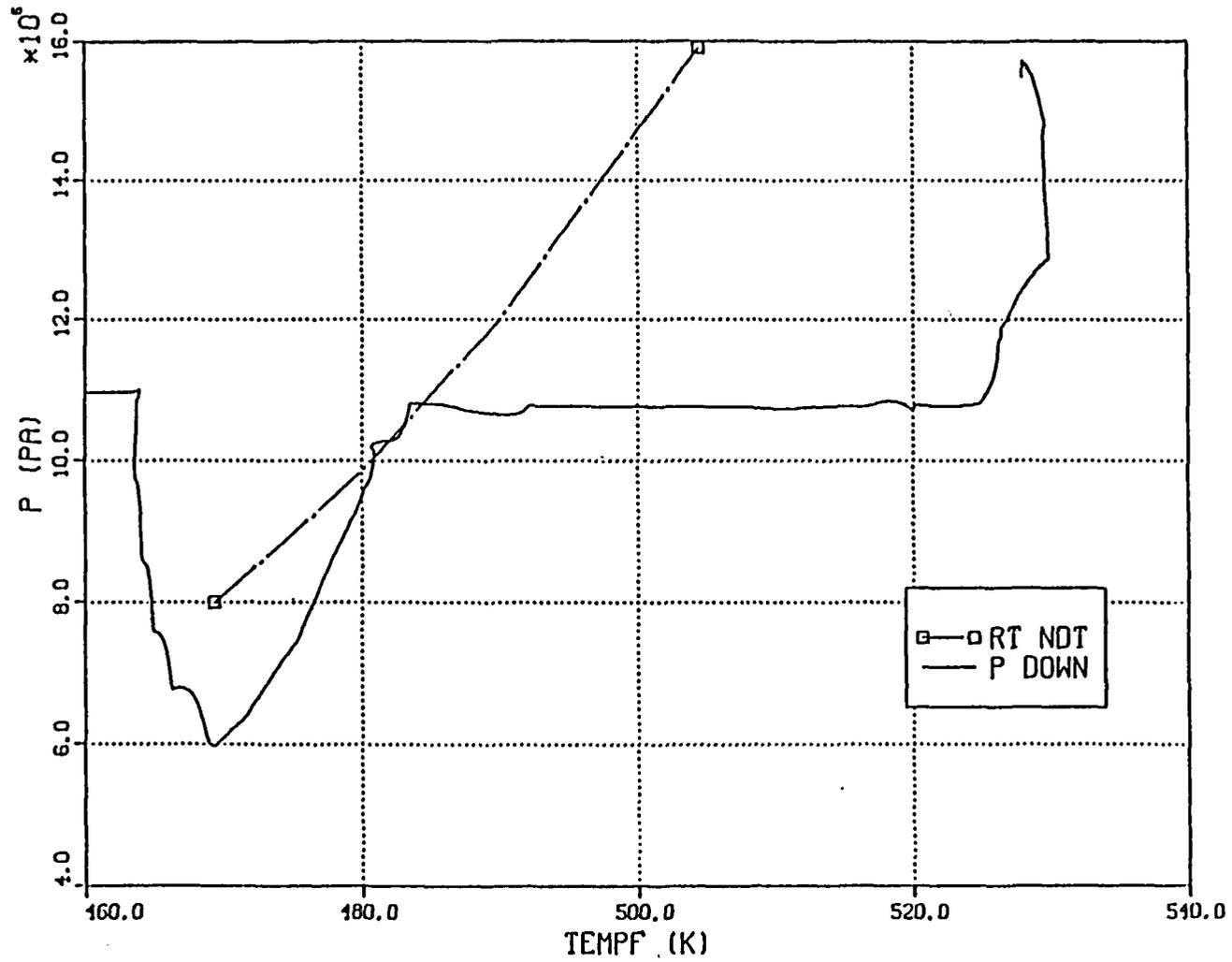
RELAP5/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM

DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

5. FIG 9.2 SUBCOOLING IN PRIMARY SYSTEM



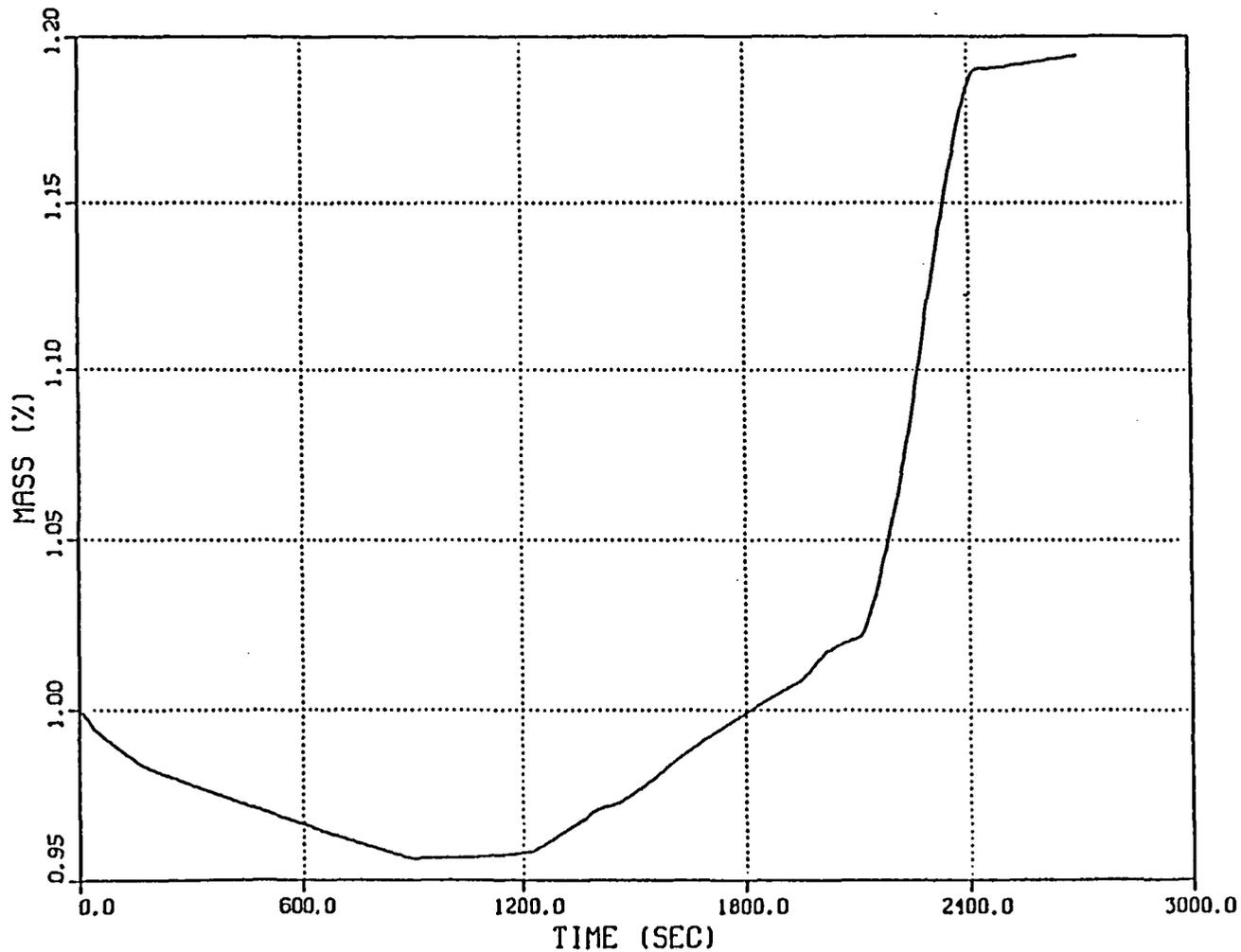
RELAP5/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM
DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.
5. FIG 9.3 PRESSURE VERSUS TEMPERATURE IN DOWNCOMER



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DOEL2 SGTR INCIDENT 25 JUNE 1979 05/09/28.

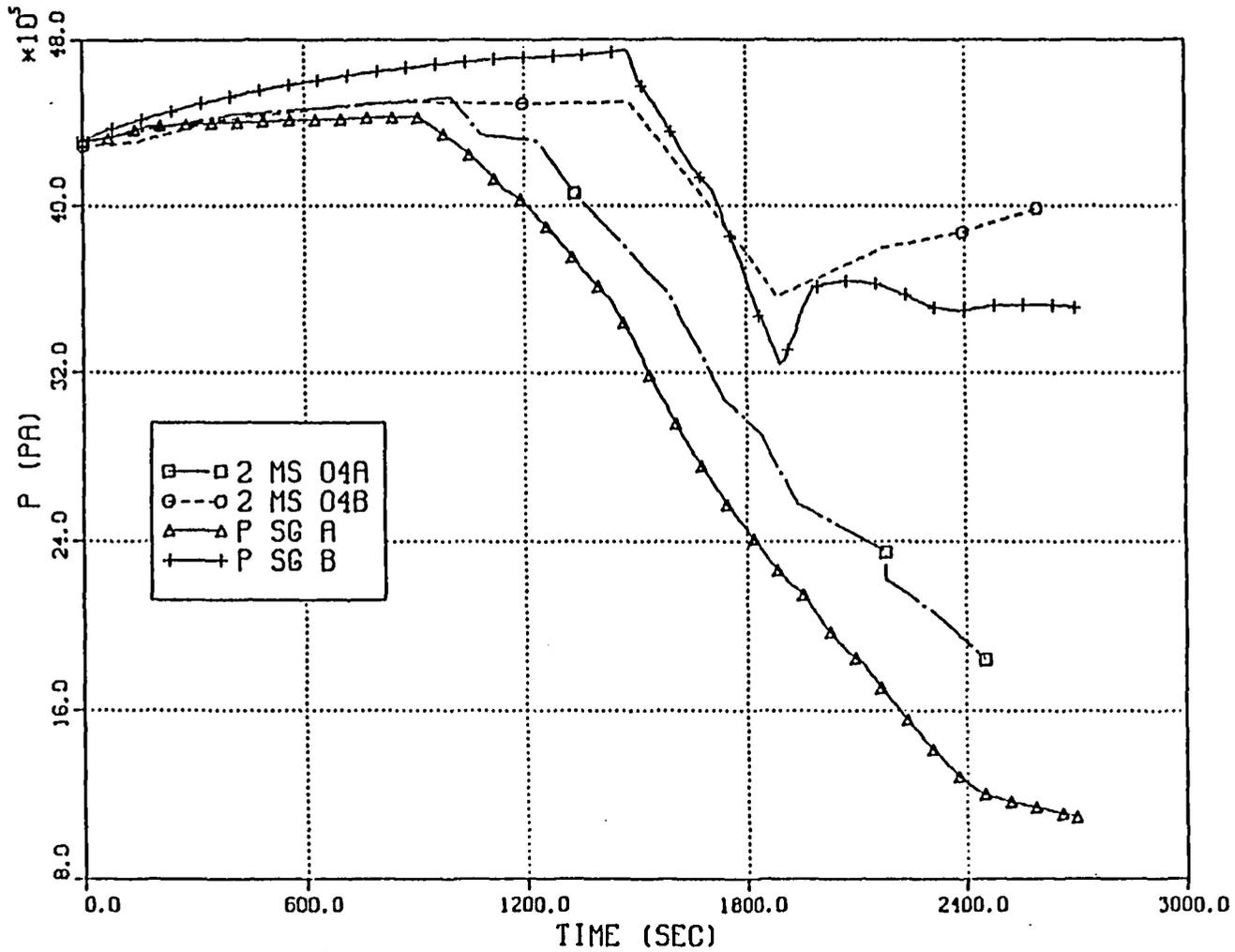
5. FIG 9.9 MASS BALANCE IN PRIMARY SYSTEM



RELAP5/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM

DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

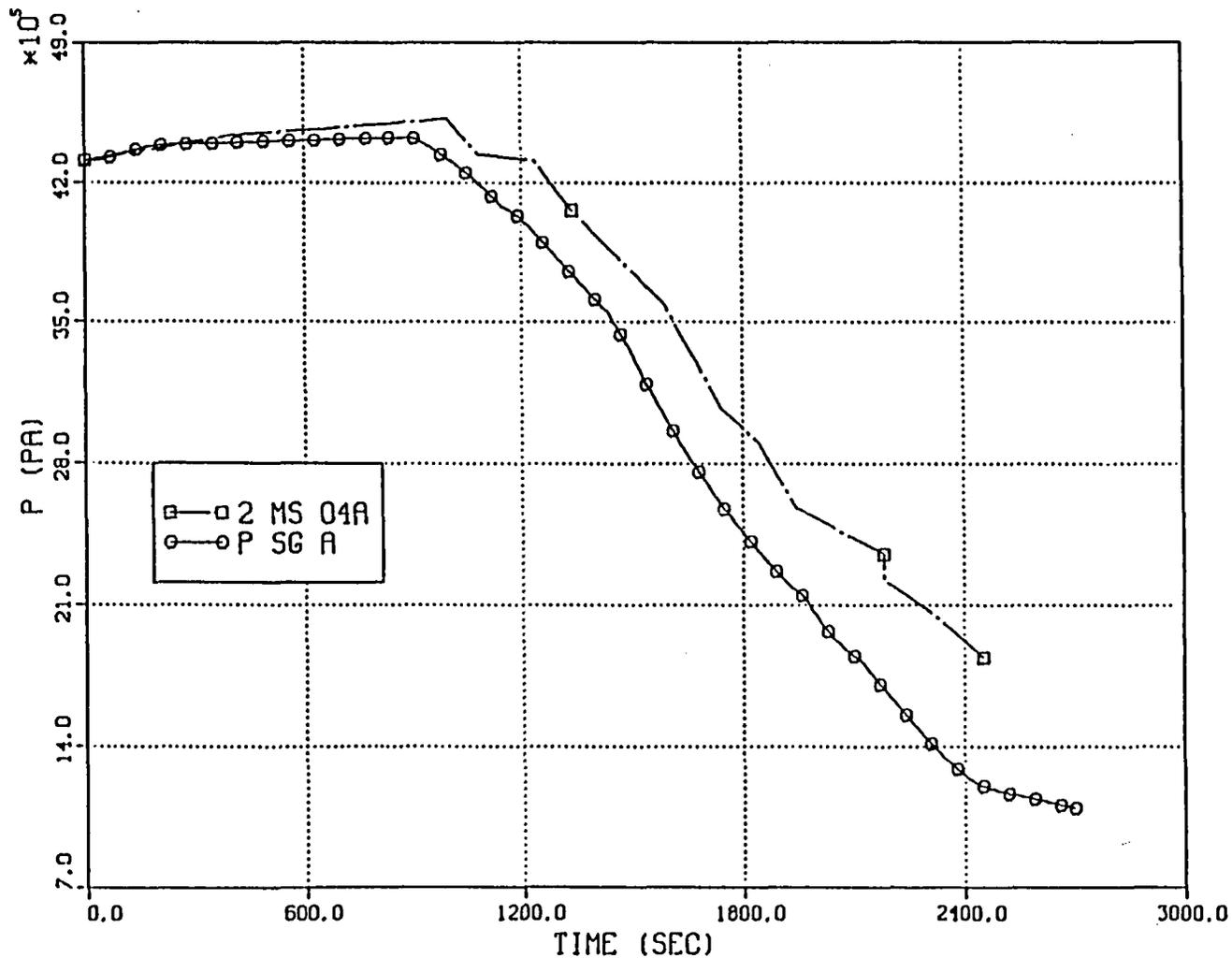
5. FIG 10.1 PRESSURE EVOLUTION IN STEAM GENERATOR (DOME)



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DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

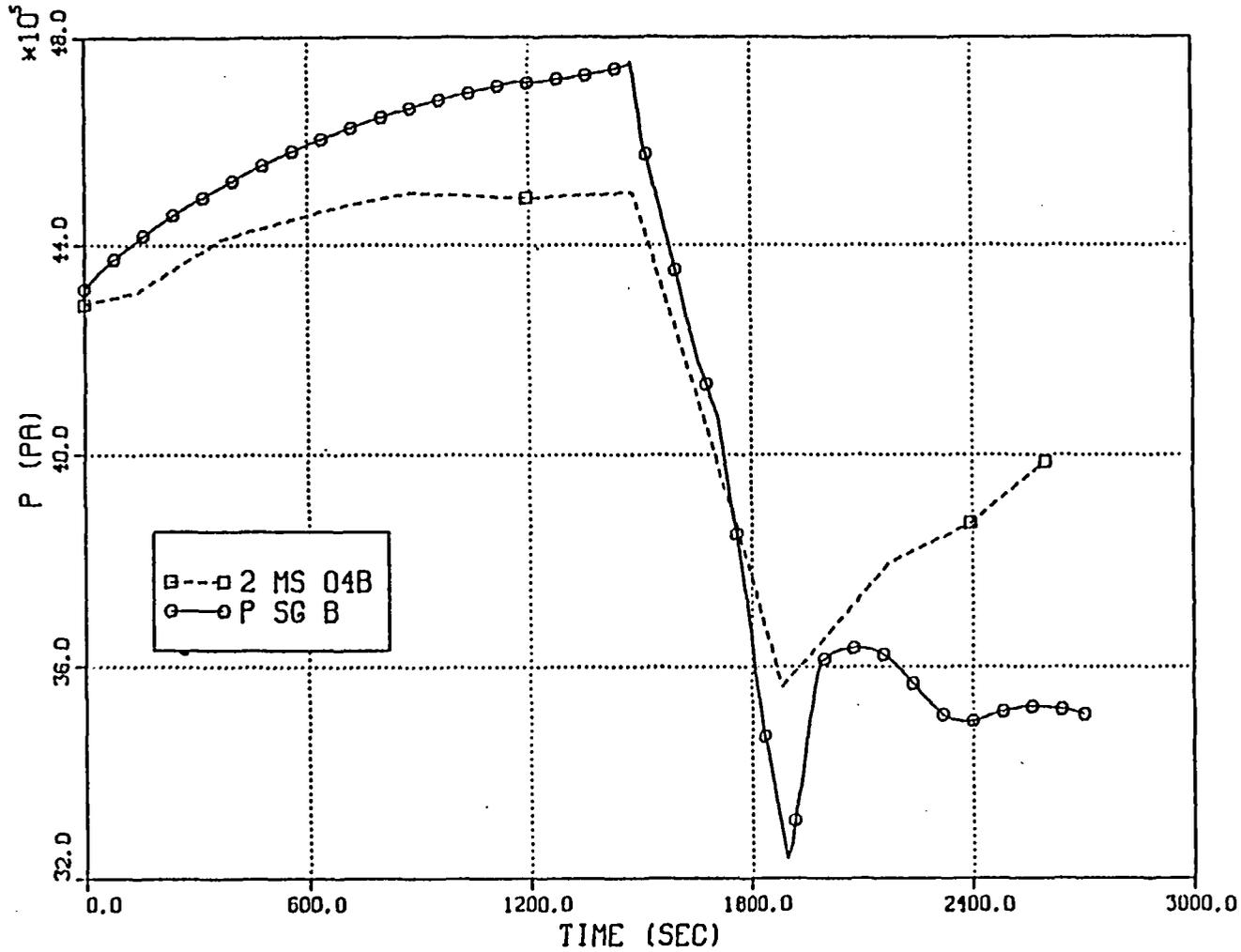
5. FIG 10.2 PRESSURE EVOLUTION IN STEAM GENERATOR A (DOME)



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DOEL2 S6TR INCIDENT 25 JUNE 1979 85/09/28.

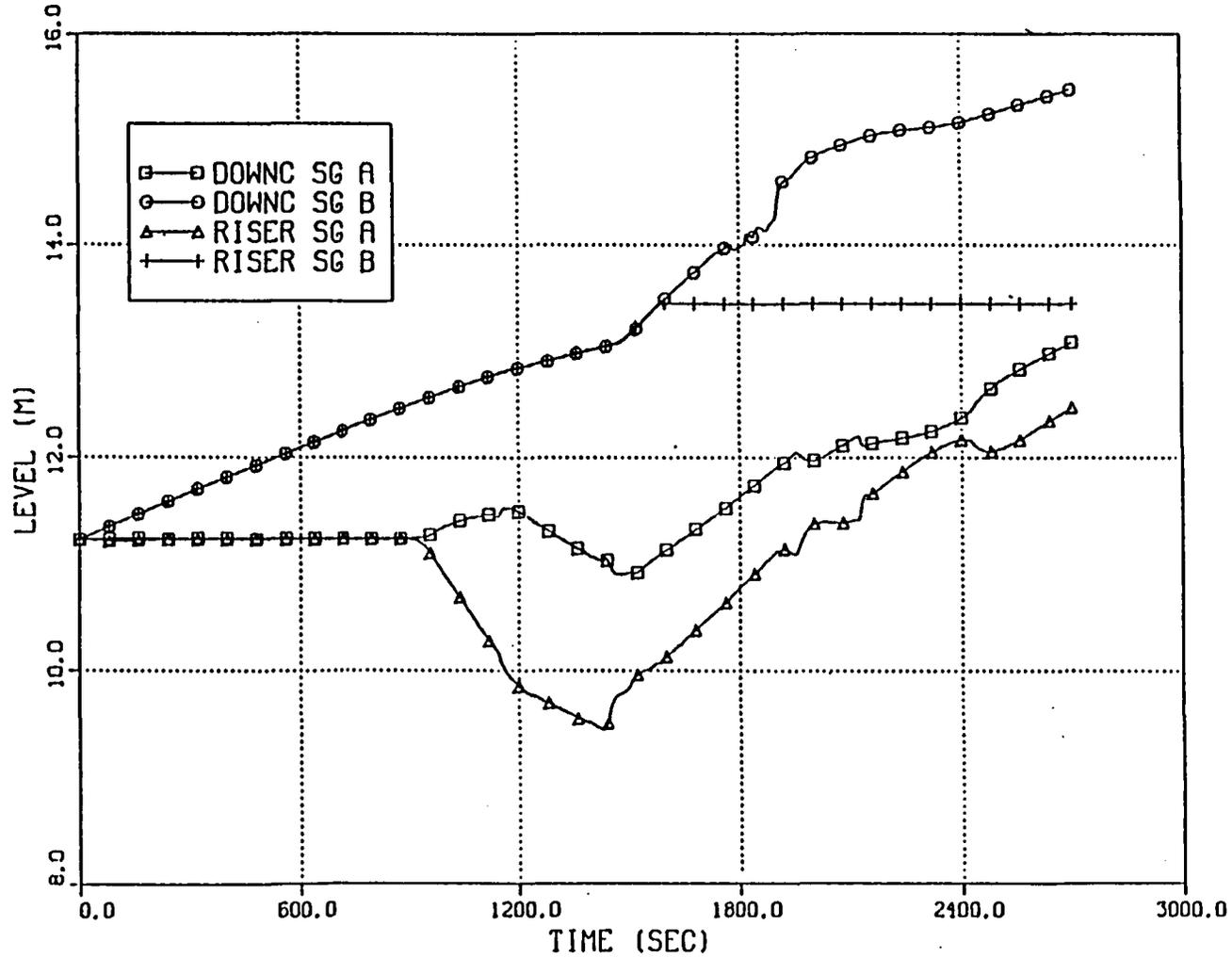
5. FIG 10.3 PRESSURE EVOLUTION IN STEAM GENERATOR B (DOME)



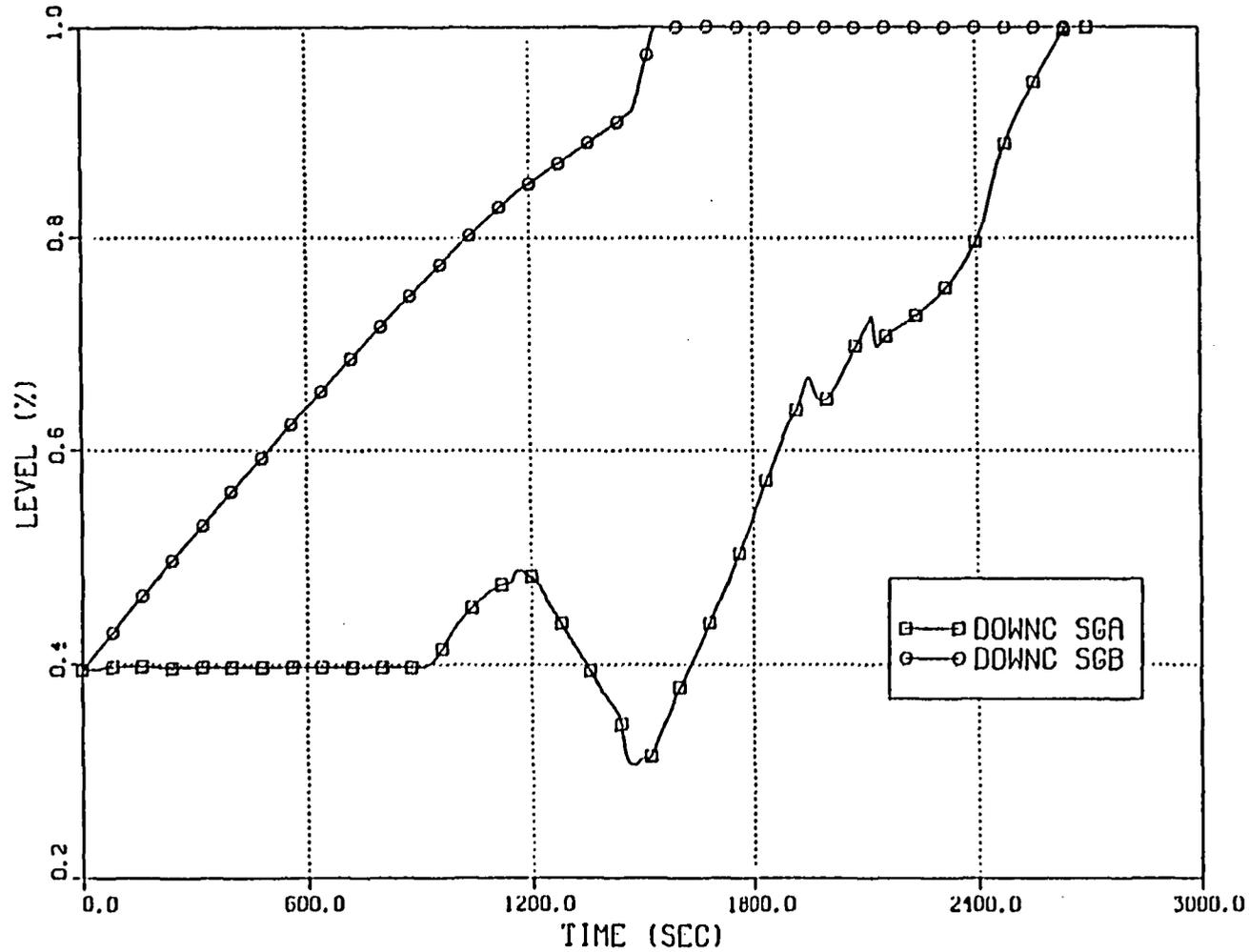
RELAP5/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM

DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

5. FIG 11.1 COLLAPSED WATER LEVEL IN STEAM GENERATORS



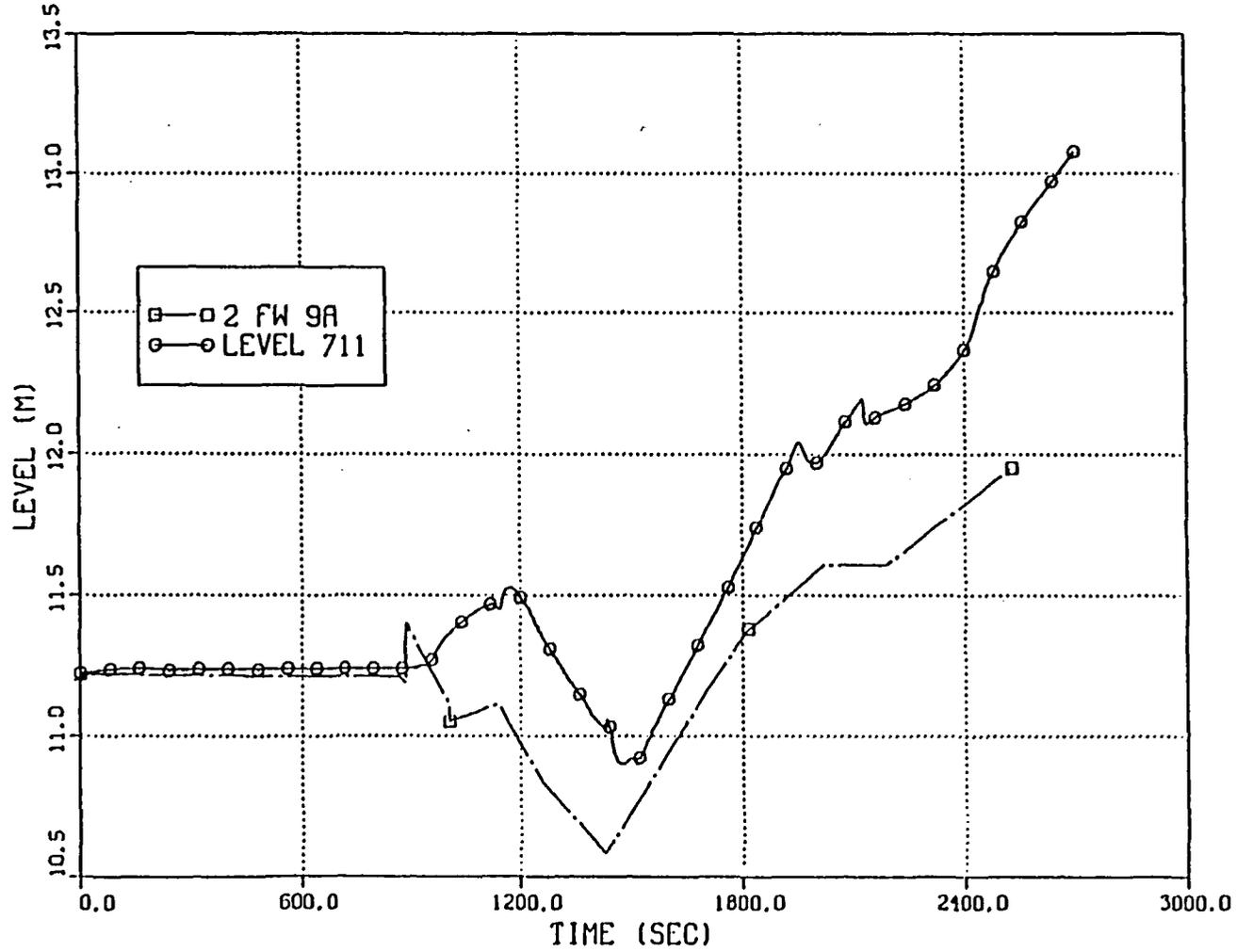
RELAP5/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM
DOEL2 SGTB INCIDENT 25 JUNE 1979 05/09/28.
5. FIG 11.5 MEASURED WATER LEVEL IN STEAM GENERATORS



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DOEL2 SGTB INCIDENT 25 JUNE 1979 85/09/28.

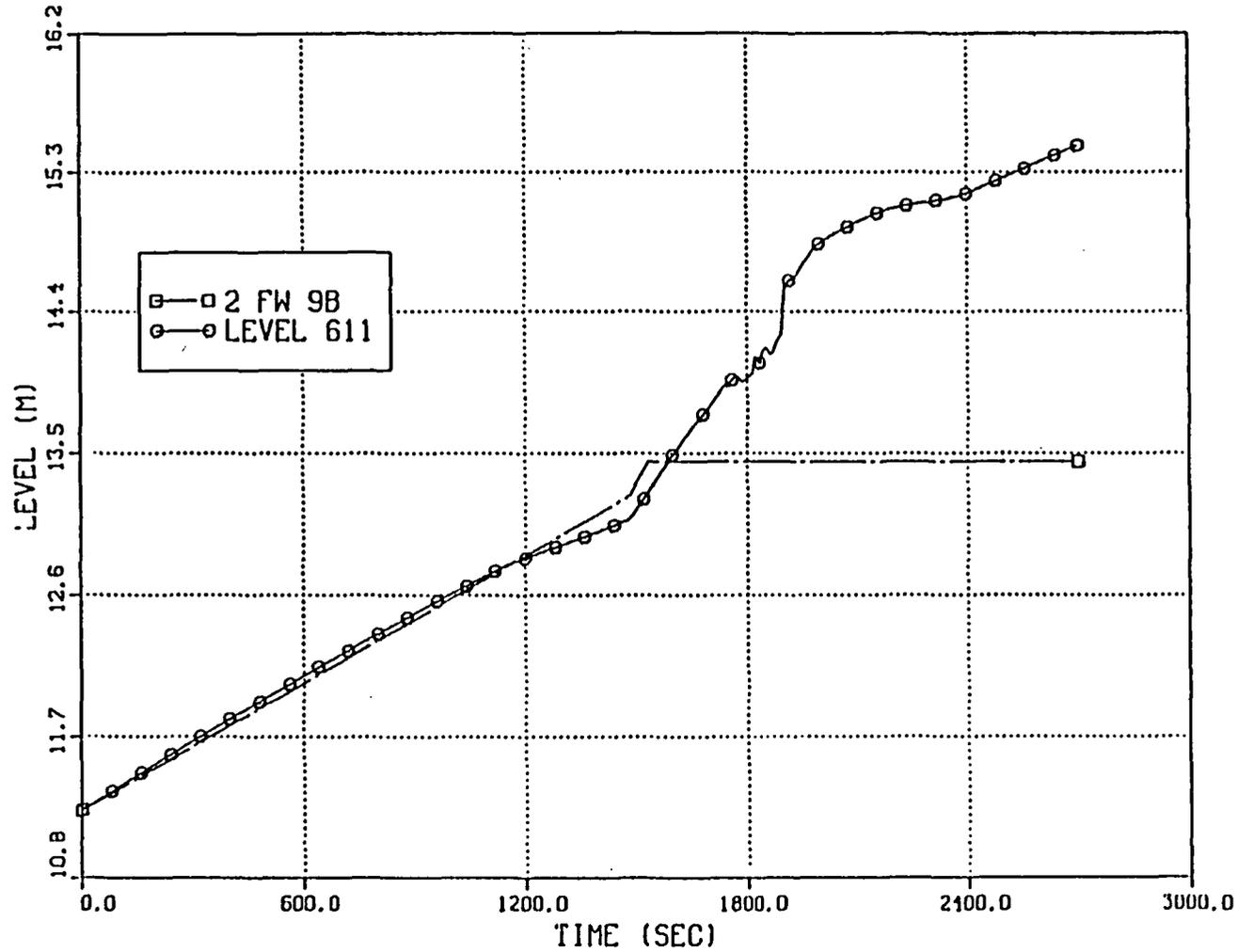
5. FIG 11.6 COLLAPSED WATER LEVEL IN STEAM GENERATOR A



RELAPS/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM

DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

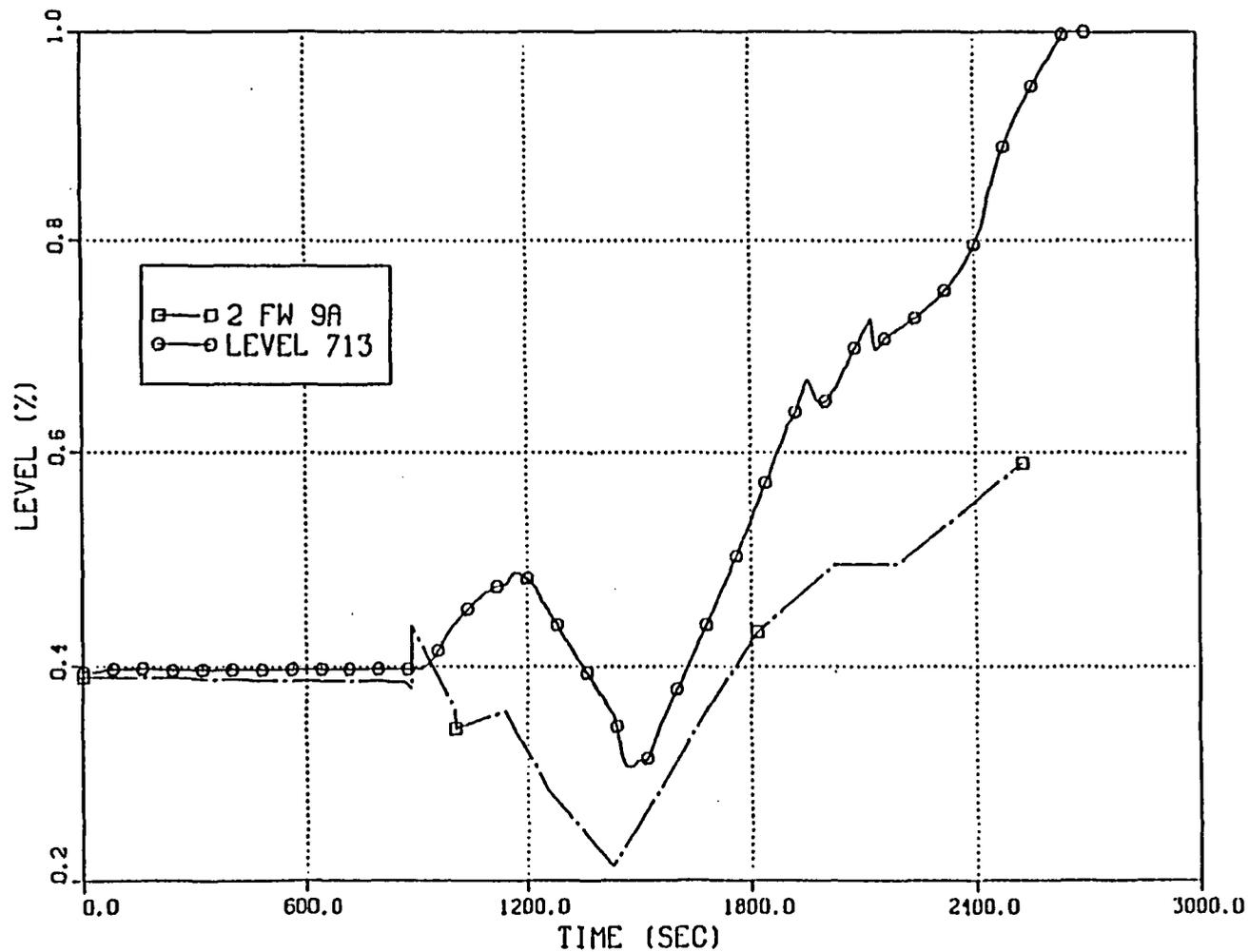
5. FIG 11.7 COLLAPSED WATER LEVEL IN STEAM GENERATOR B



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DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

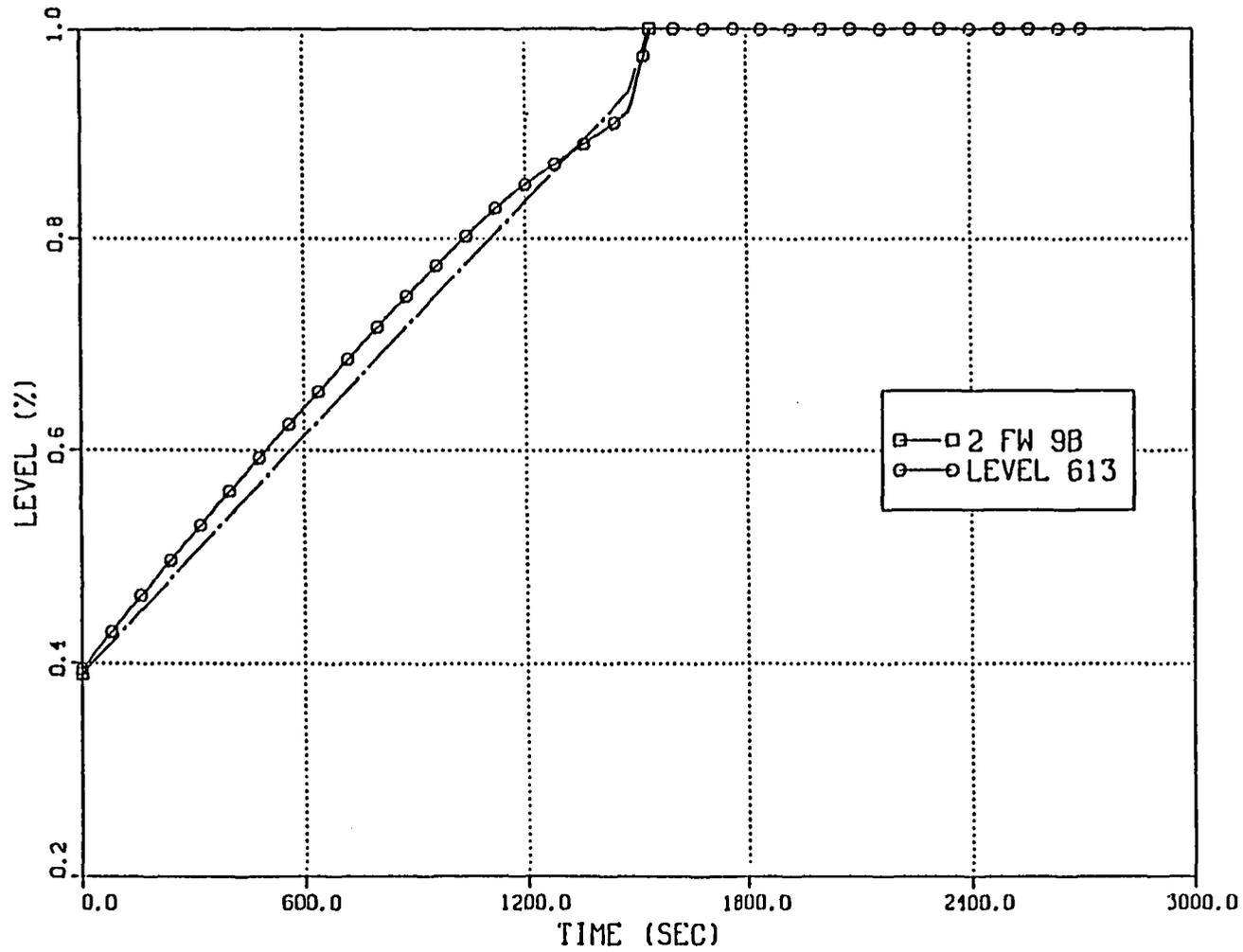
5. FIG 11.8 MEASURED WATER LEVEL IN STEAM GENERATOR A



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DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

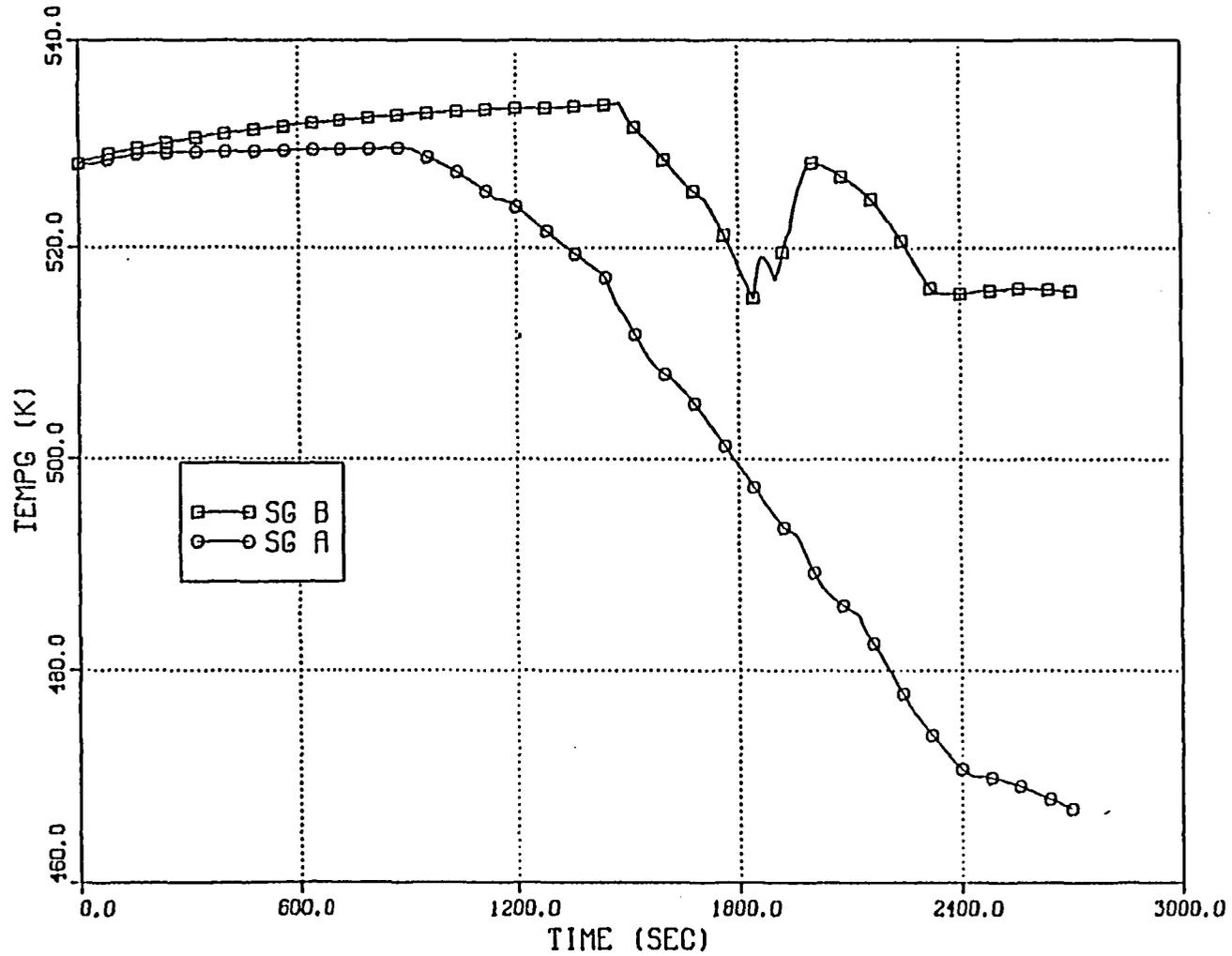
5. FIG 11.9 MEASURED WATER LEVEL IN STEAM GENERATOR B



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DOEL2 SGTB INCIDENT 25 JUNE 1979 85/09/28.

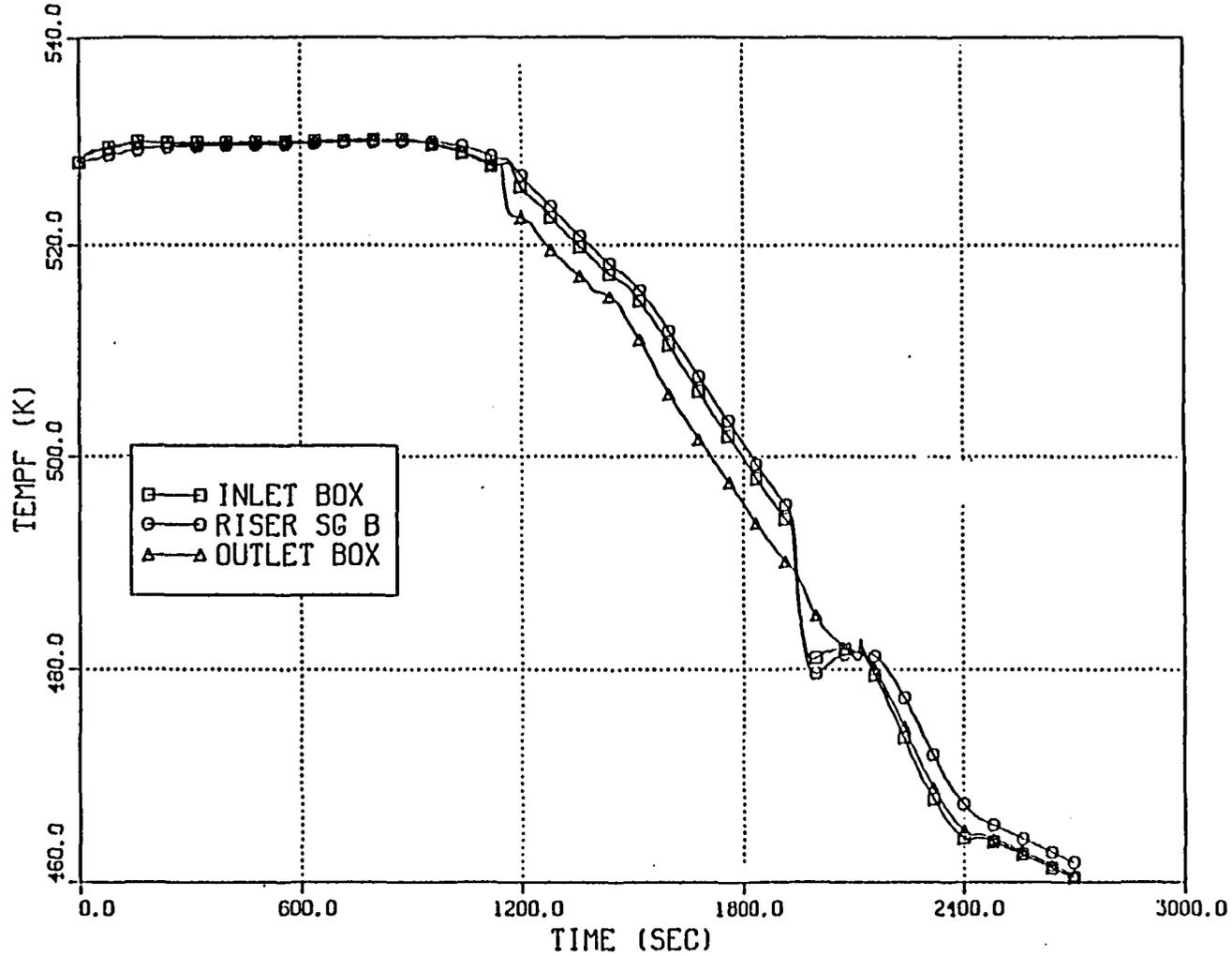
5. FIG 12.1 STEAM TEMPERATURE IN STEAM GENERATORS(DOME)



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DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

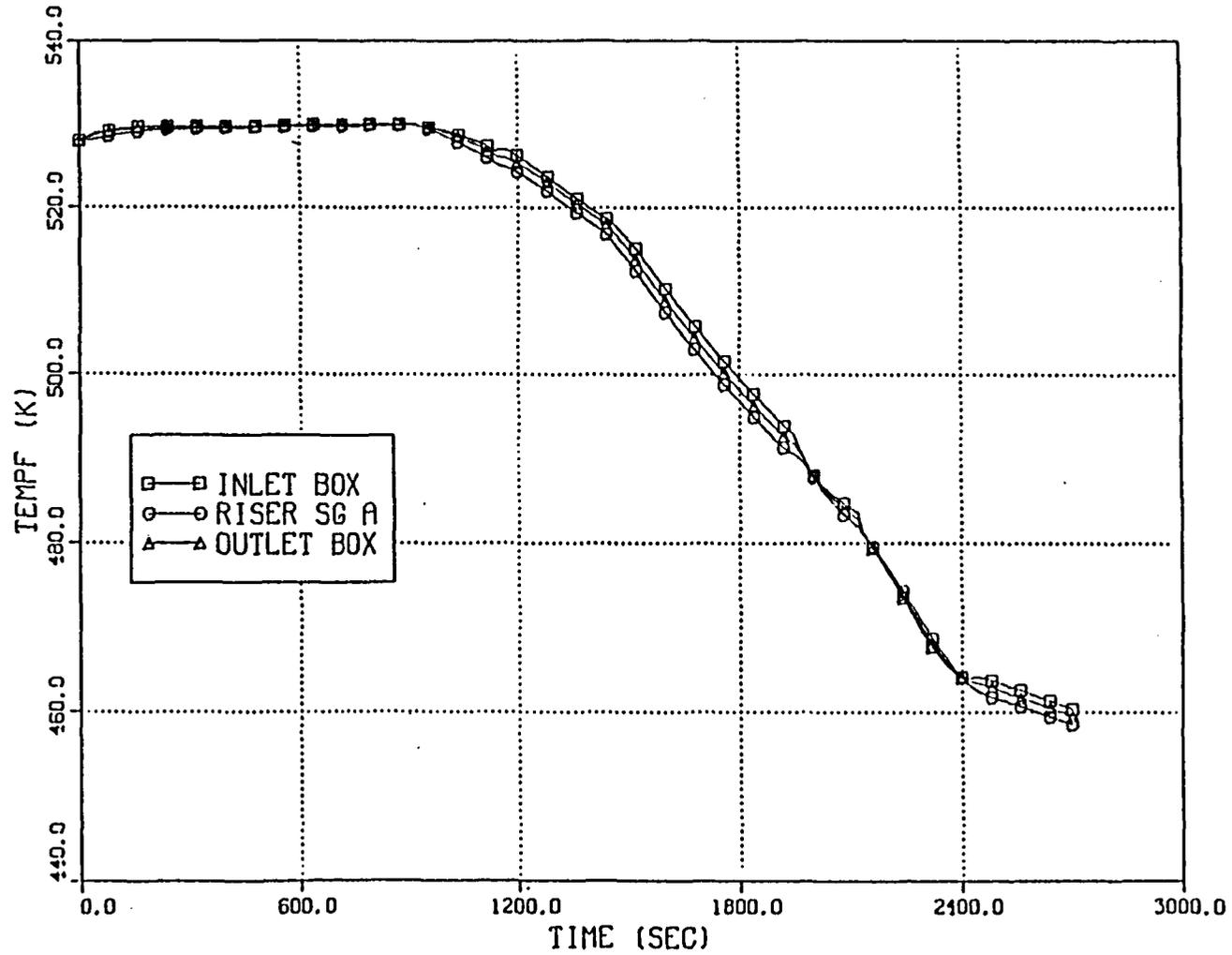
5. FIG 12.2 LOOP B AND SG B TEMPERATURES



RELAP5/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM

DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28;

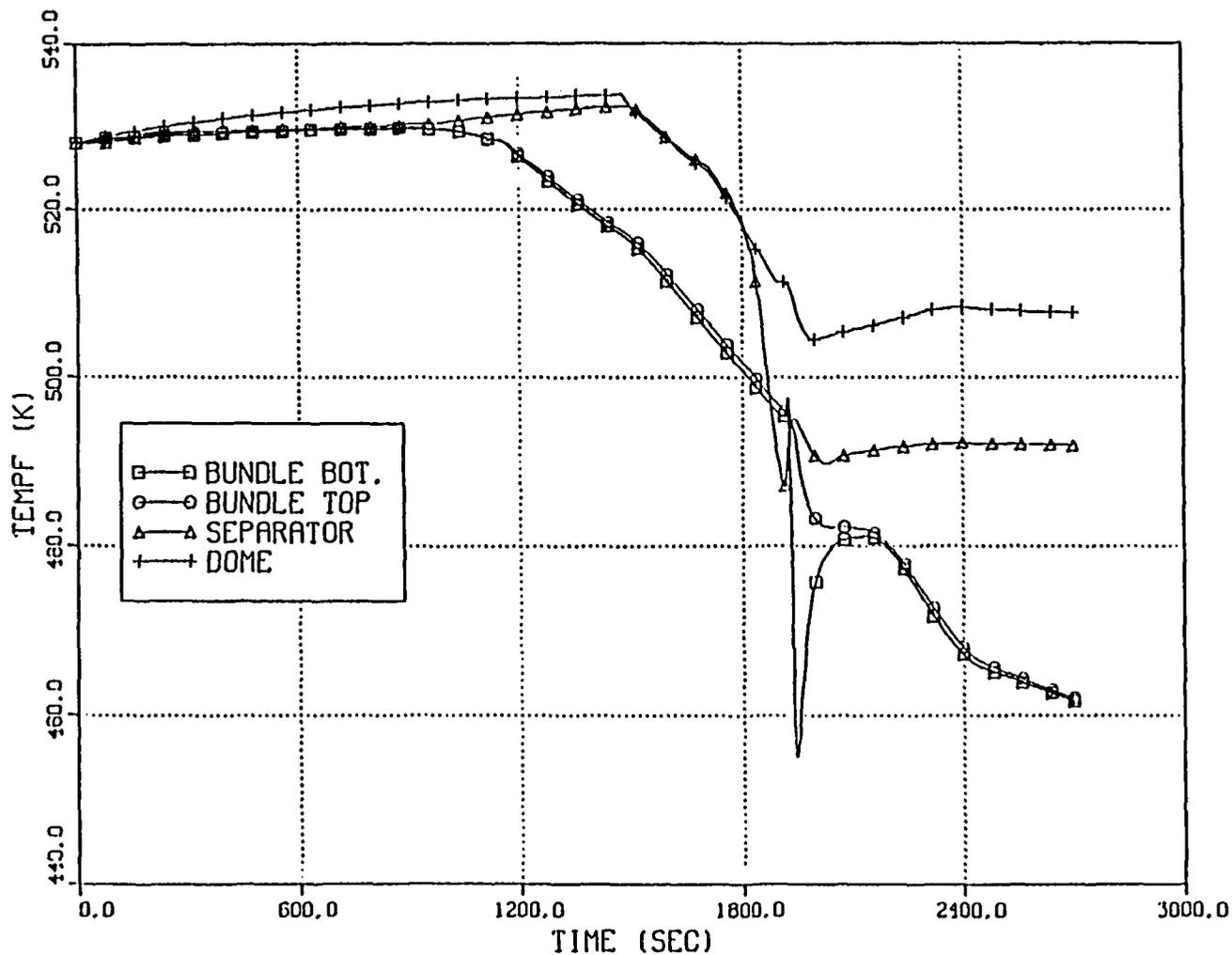
5. FIG 12.3 LOOP A AND SG A TEMPERATURES



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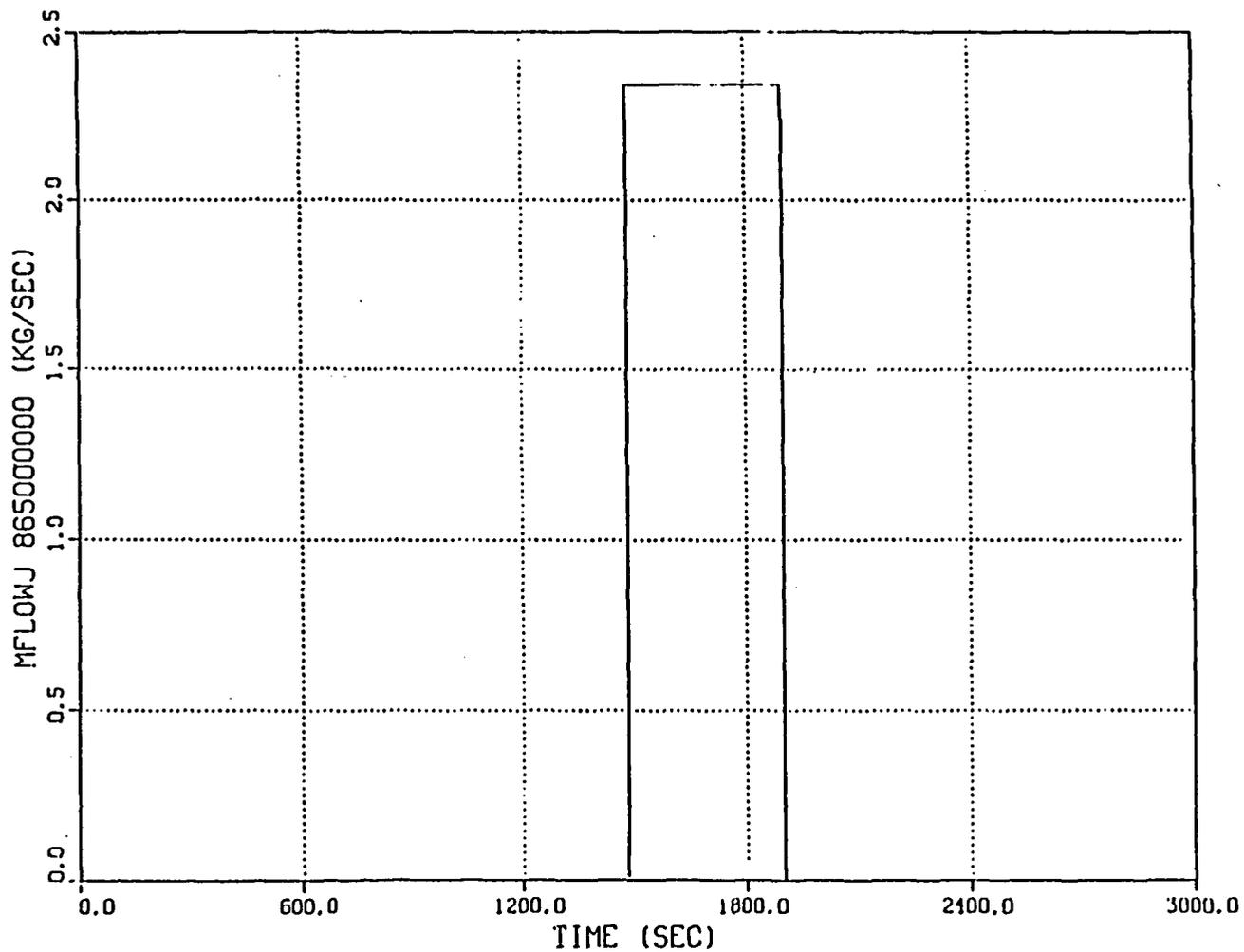
5. FIG 12.4 TEMPERATURE STRATIFICATION IN SG B



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DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

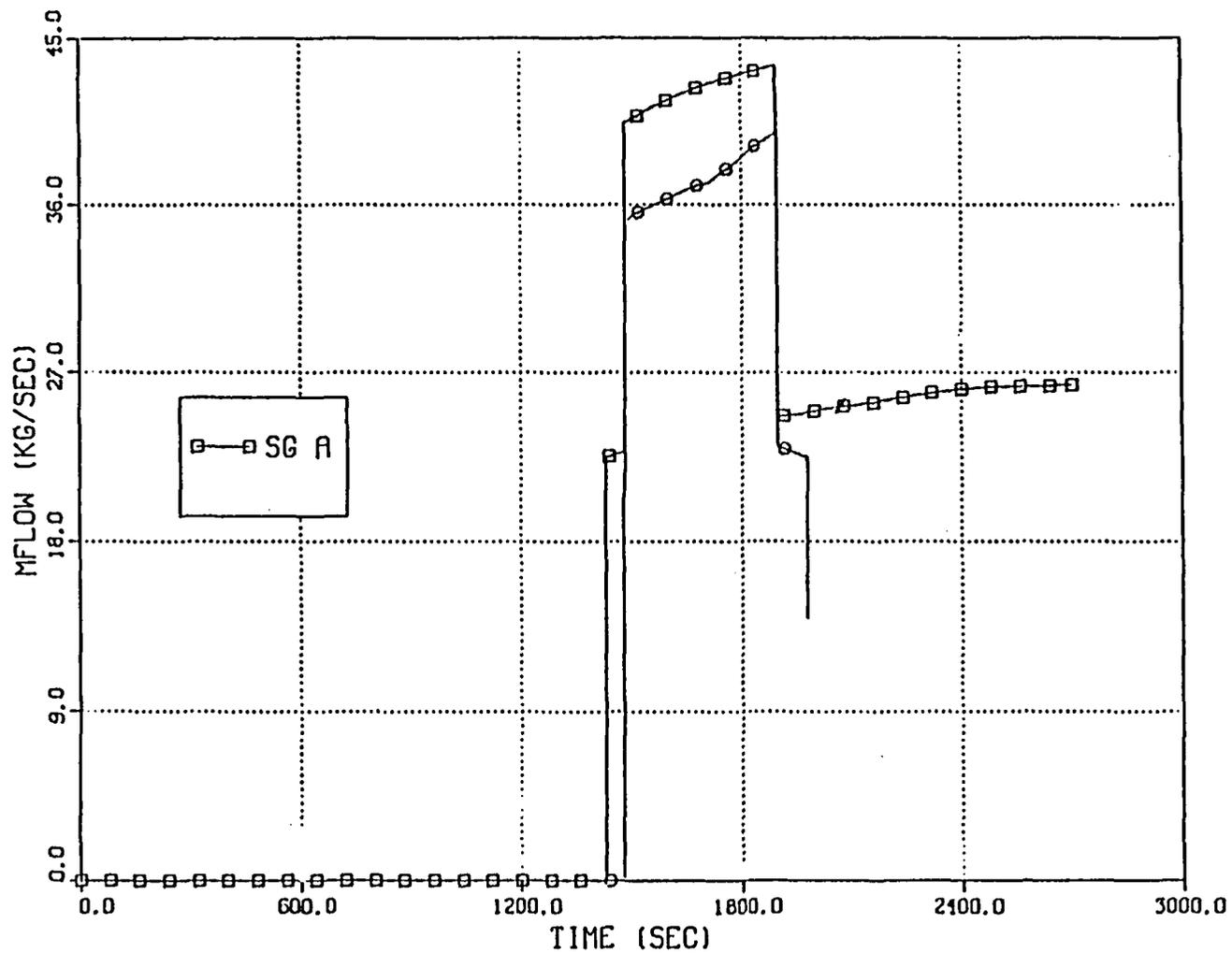
5. FIG 14.7 STEAM MASS FLOW RATE FROM SGL TO TURBOPUMP



RELAP5/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM

DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

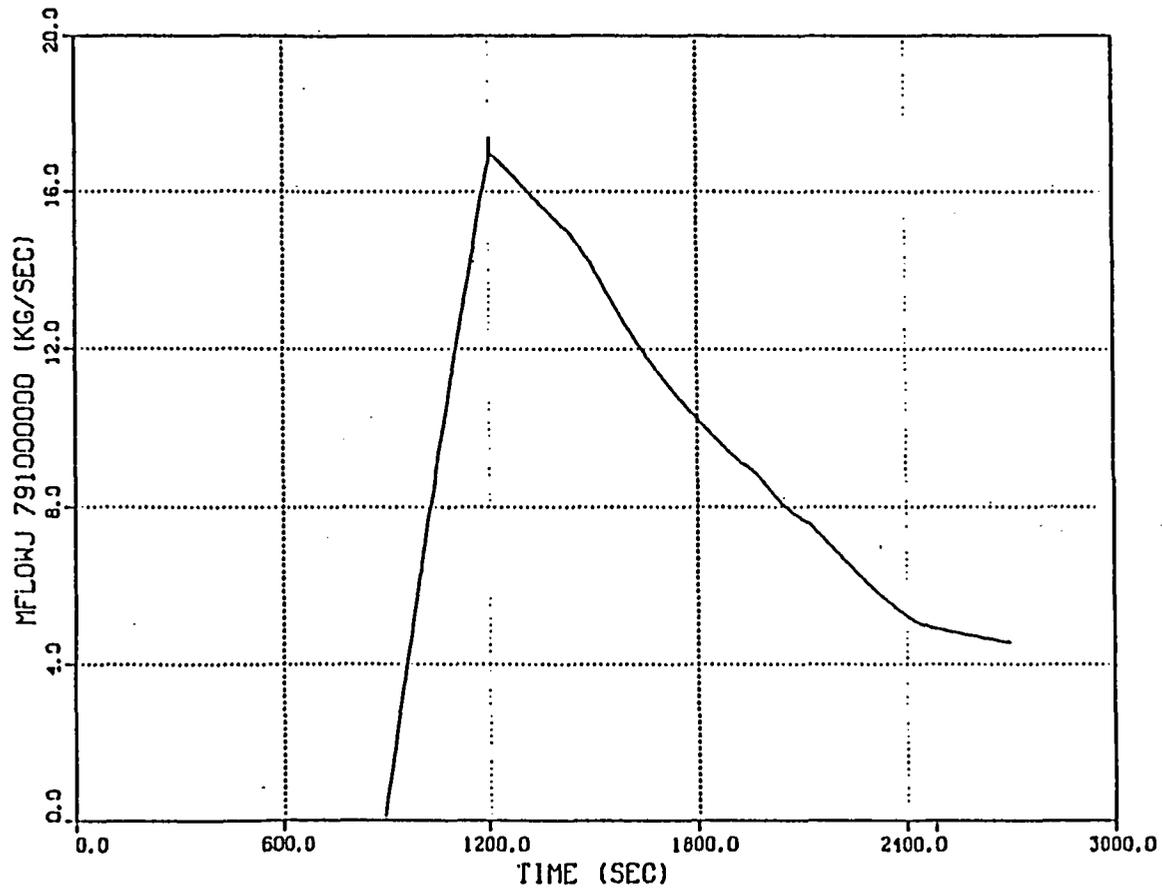
5. FIG 14.8 FW AND AFW TO STEAM GENERATOR



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DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

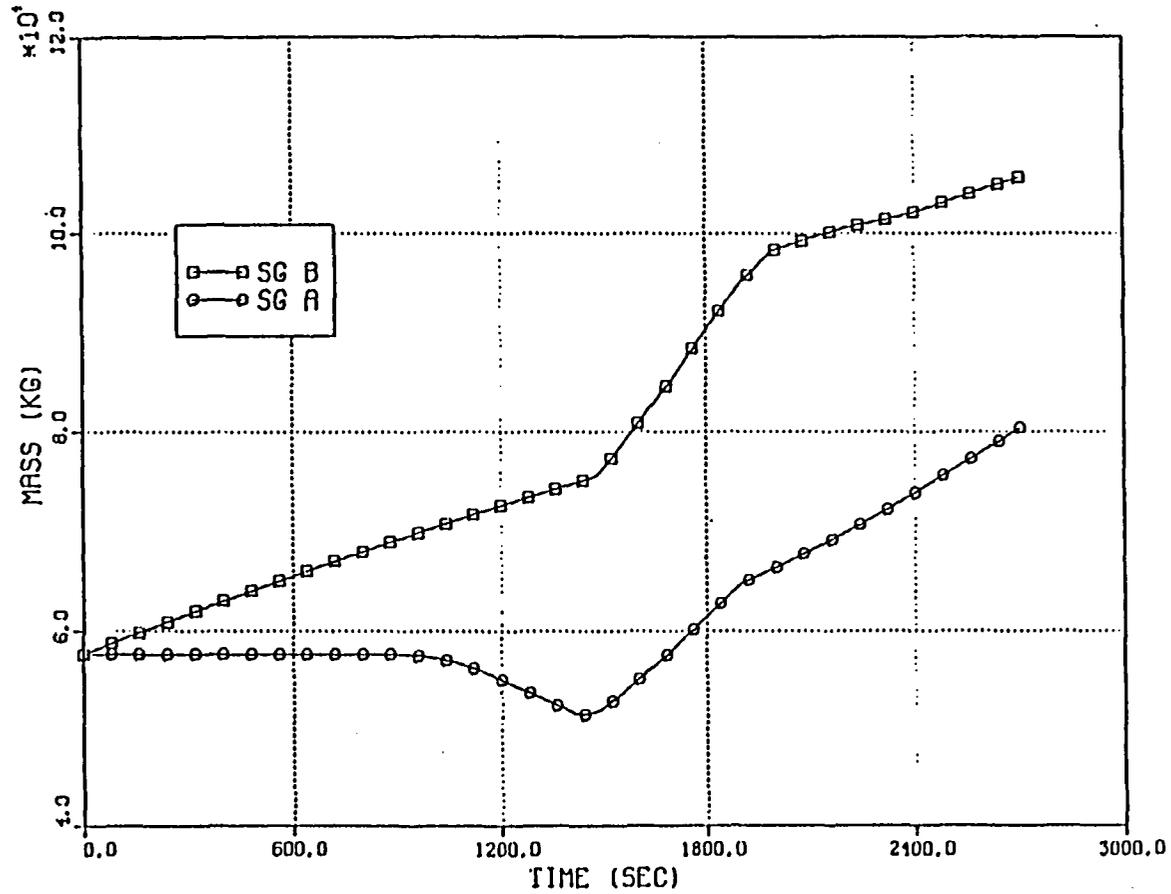
5. FIG 14.9 SG A RELIEF VALVE MASS FLOW RATE



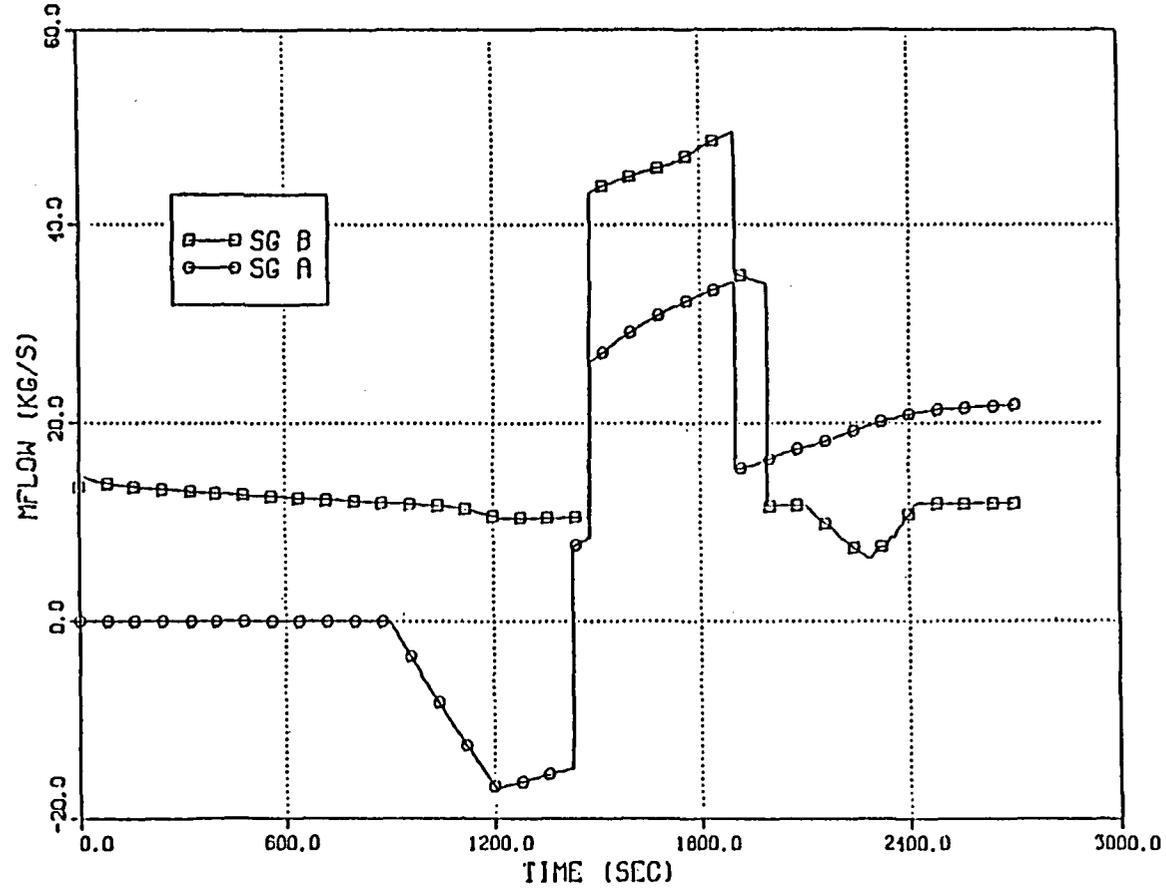
RELAPS/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM

DOEL2 SCTR INCIDENT 25 JUNE 1979 05/09/28.

5. FIG 15.1 STEAM GENERATORS MASS EVOLUTION



RELAP5/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM
DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/20.
5. FIG 15.2 SG'S MASS ADDITION AND DEPLETION RATES



6. DISCUSSION OF THE BASE CASE RESULTS

From the comparison between available plant data and calculated data we can state that the code yields in general agreeable results as far as the trends are concerned. A quantitative code assessment however reveals some serious code deficiencies which will be discussed separately.

6.1. Water level discrepancy in intact S.G. at start of cooldown

Fig. 5.11.8 manifests a level discrepancy at the time the intact S.G. steam relief valve is actuated to initiate a cooldown.

There exists indeed a large uncertainty in the time of the cooldown initiation and the opening rate of the relief valve. A sensitivity study performed earlier, (and discussed in chapter 7.1) by which the valve was opened instantaneously showed a very large level swell which clearly deviated from measured data. Based on this evidence it was decided to program a linear opening rate of the relief valve between 900 and 1200 seconds for this study.

Fig. 5.11.8. shows that the measured water level manifested a shortlived level spike which could be indicative of an actual level swell phenomenon. The irregularities observed could also be due to unknown operator interventions. Debugging the major edit at 1000 and 1200 s, manifests following phenomena:

- . the flowregime in the SGA riser is of the BBY or SLUG type;
- . the liquid in the riser is entrained by the vapour flow, such that the liquid velocity varies between 1/3 and 1/2 the vapour velocity (around 0.5 m/s), indicative of a high interface friction.
- . the FIJ increases from bottom (720) of riser to top (722) from about 3000 to 14000, and is suddenly falling to low values at the separator outlet junction.

This produces a liquid fall back into the separator volume where water accumulates, hence leading to a large ΔP level

as the smallband level taps are located just above and below the separator.

Even if the precision in the recorded levels is not known, the calculated trend diverges from the measured trend most probably due to excessive level swell in the riser region combined with liquid fallback in the separator region. This would be indicative of too high an interphase friction in the S.G. riser section for the prevailing flow regime (SLUG).

Since the water level measurement in a power plant is used for plant control and also for plant protection, it follows that numerical discrepancies for predicting SG water level could have a large impact on the predicted behaviour of a power plant, and in the timing of the actuation of emergency safeguard features (e.g. auxiliary feedwater).

6.2. Excessive cooldown between 1500 and 2400 seconds

From the comparison between measured and calculated temperatures in the primary loops (fig. 5.5.4., 5.5.5., 5.5.6. and 5.5.7.), the calculated cooldown rate starts diverging from the measured cooldown rate around 1500 sec, which corresponds with the start of the turbine driven auxiliary feedwater pump, whereby steam is derived from the affected S.G. to drive the turbine driven auxiliary feedwater pump, and extra feedwater is fed to both steam generators.

Due to uncertainties in the turbine driven auxiliary feedwater pump characteristics (steam flow, auxiliary feedwater flow), it is not warranted to look for code model deficiencies which may explain this discrepancy.

During the period wherein the turbine driven auxiliary feedwater pump is operational, the affected S.G. acts as a heat source (fig. 5.12.2), whereby about 7.7 MW of heat is extracted from the S.G. riser volumes, via the SG U tube bundle into the primary system. Notice that due to the availability of only 1 primary coolant pump in the intact

loop, a reversed flow exists in the affected loop and cold SI water injected downstream of the stopped coolant pump enters directly into the affected SG, leading to an enhanced cooldown of the riser.

Also the location of the break in the top of the riser provides a cooldown by injecting colder water from the primary. Both effects lead to a strong gravity head in the riser.

On the downcomer side, one observes a strong heat transfer by conduction between volumes 622 and 612/613 (about 3 MW) and a large structural heat input in the injection volumes 612/613 which tends to heat up the cold AF water.

The net result is a reversed buoyancy effect which tends to drive a reversed circulation in the affected SG ($K_R = 3 \times 1000$).

When the turbine driven auxiliary feedwater pump is switched off, the steam flow, taken from the affected SG, is halted and this stops the strong depressurization in this SG. Also the temperature drop is stopped whereby the structural heat (the driving force for the reverse circulation) is strongly reduced.

Between 1900 and 1992, cold auxiliary feedwater is still injected in the affected SG by the motordriven pump. Both effects lead to a reversal in the natural circulation from initially - 11 kg/s at 1900 s, to over 800 kg/s at 1940 s and dropping down to 160 kg/s at 2000 s.

At 1900, the fluid temperature in the volumes surrounding the feedwaterline (vols 611, 612, 613) is about 20° below the bundle fluid temperature. When the circulation sets in, in the normal direction, this large inventory of cooler fluid, combined with a continuous injection of cold AFW from the motor driven pump (flowrate = 23 kg/s) produces a strong cooldown

in the riser and constitutes an abnormal large heat sink for the RCS (see fig. 5.7.3) of max. 50 MW.

The strong reversal in the temperature at bottom of the SG is illustrated in fig. 5.12.4, showing the effect of an overcooling by more than 20°C when the natural circulation sets in.

From the discussion of a parametric study (chapter 7.2.) it is believed that this anomaly is caused principally by a wrong choice of the forward and reverse form losses in the S.G. downcomer.

In this study, very large reverse form losses and low forward form losses may act as a leaky diode where during periods of reversed buoyancy colder water leaks slowly in the reverse direction without mixing with hotter fluid on top of the downcomer. When the buoyancy changes over to the normal circulation, excessive natural circulation flowrates are observed leading to the observed phenomena.

Since it is known that RELAP-5 tends to exaggerate the natural circulation flows, it may be recommended to select suitable form loss factors both for direct and reversed circulation (e.g. $K_f = K_r = 200$ in downcomer junction) in order to allow a larger reversed flow, and hence better mixing, and a reduction of the forward flow which should lead to smoother temperature transitions.

The choice of the form loss factors should however be consistent with plant values in order to lead to the known recirculation ratio at various power levels.

The effect of this apparent anomaly propagates back into the RCS where it aggravates the temperature discrepancy in the cold legs of the primary loops.

When this colder water is used as spray in the pressuriser at 2100 seconds, the excessive spray subcooling must produce an excessive depressurization as shown in fig. 5.4.1.

(discrepancy about 9 bar between calculated and measured

pressure peak), which in turn increases the cold HPSI flowrate (fig. 5.6.7.) and yields a positive feedback to the overcooling.

This example shows how a code anomaly can propagate back from one S.G. to the primary and lead to a large discrepancy which maintains itself through the existence of a positive feedback (e.g. pressure - SI flow).

6.3. Vapour condensation anomaly in affected S.G.

When the affected SG is completely bottled-up after tripping the AFW turbo and motor-driven pump at 1992 s, a large fluid temperature stratification exists in this SG (fig. 5.12.4) and the difference between fluid temperature and vapour temperature (fig. 5.12.1) in the SG dome levels out at about 7°C.

From the pressure history (fig. 7.10.1) one observes a sudden increase in pressure when the turbine driven auxiliary feedwater pump steam valve is closed at 1900 s, but levels off at a pressure of about 36 bar around 2000 s, and has a tendency to decrease afterwards while the recorded pressure keeps rising.

In order to explain this anomaly, one should evaluate the vapour generation/condensation phenomena under such high temperature disequilibrium, for which fig. 6.1 is a good illustration of the phenomena occurring the SG dome. This figure illustrates a vapour generation rate increasing sharply at 1830 s, when the water level crosses the lower boundary of the upper dome volume. Since a forced steam flow of 2.34 kg/s (fig. 5.14.7) is derived to the turbine driven auxiliary feedwater pump (requiring a vapour generation rate of 0.089 kg/m³,s) the actual vapour generation rate is much lower (fig. 6.1) leading to a net vapour mass depletion and a resulting accelerated pressure drop (see fig 5.10.1).

When the steamline to the turbine driven auxiliary feedwater pump is closed at 1900 s, the vapour generation continues for a while leading to a sharp pressure increase, until the vapour generation reverses to a non negligible vapour condensation at 1960 sec. of $0.009 \text{ kg/m}^3 \cdot \text{s}$ (or about 0.237 kg/s).

This condensation of vapour combined with a net mass addition to the affected SG from the break, leads to a stagnation in the pressure (or even a slight pressure decrease) in contrast to the recorded pressure increase.

This anomaly is likely due to an excessive vapour condensation rate in the dome volume of superheated vapour (523°K) on subcooled fluid (507°K) for a saturation temperature of 517°K (at 2200 s).

From an energy balance in volume 630 at 2200 s, one obtains a vapour-water condensation flux of 436 kW resulting in a vapour-water condensation heat transfer coefficient of about $2100 \text{ W/m}^2 \cdot ^\circ\text{C}$. Such excessive heat transfer coefficient is calculated for a prevailing flow regime "SLUG", while the volume liquid velocity is negligible ($\approx 6 \cdot 10^{-4} \text{ m/s}$) and the volume vapour velocity also is negligible ($\approx 3 \cdot 10^{-4} \text{ m/s}$). For such velocities a "STRATIFIED" regime should exist which would probably have resulted in much lower vapour condensation rates.

Hence, the pressure anomaly can be traced via a condensation anomaly for a wrong flow regime for this volume.

Finally it should be emphasized that an underestimation of the pressure in the affected S.G. could have an impact on the postulated timing of automatic systems and operator interventions:

a higher S.G. pressure will lead to an earlier isolation of the break flow from primary to secondary and an earlier switch-over to post-SGTR cooldown procedures.

a higher SG pressure may lead to actuation of the SG safety valves, with the danger of failing to close when water should be released.

CONCLUSION :

Two important shortcomings of the code are evident from this analysis :

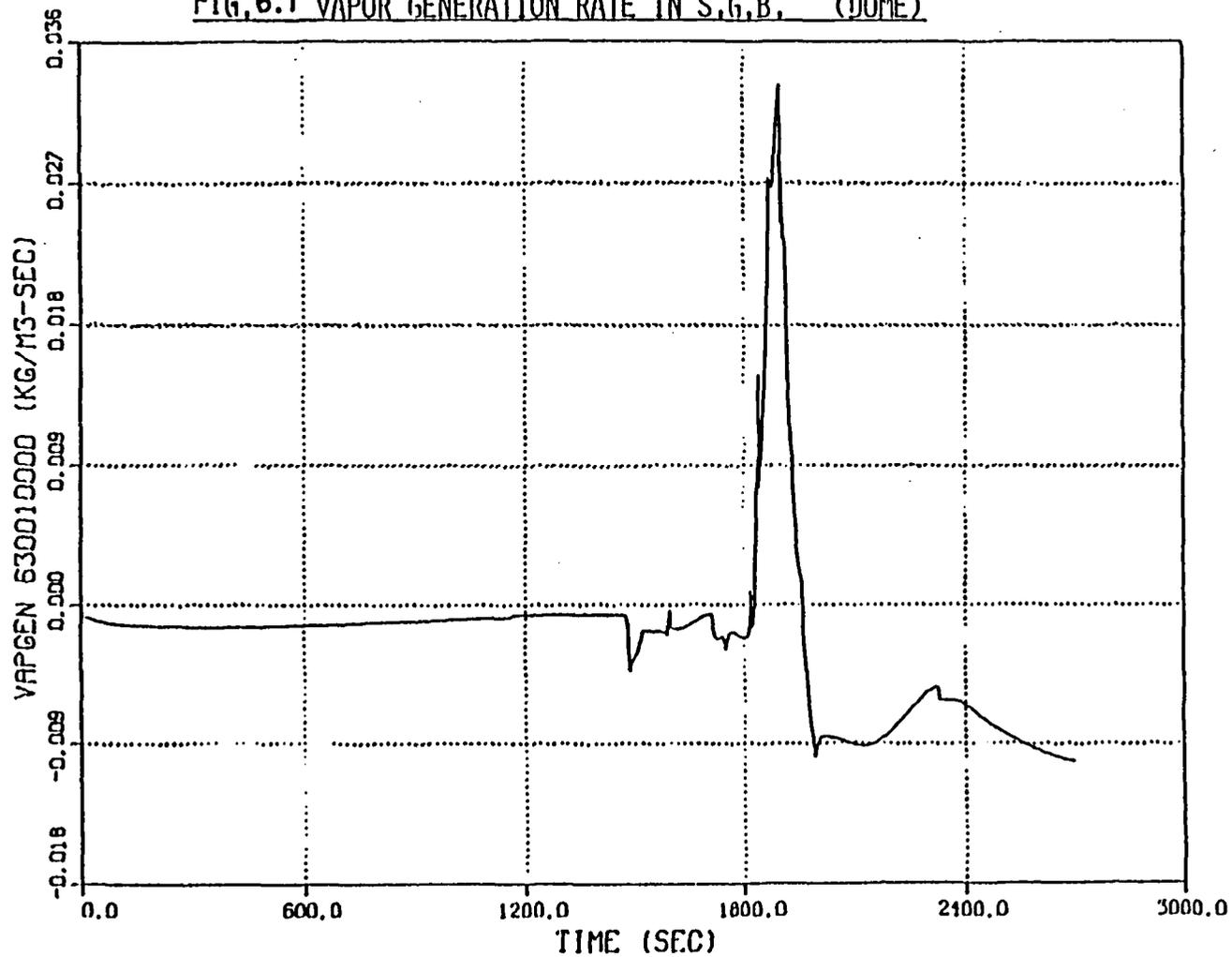
- a. Excessive interphase drag in the S.G. riser is observed under certain conditions.
- b. Excessive heat and mass transfer for direct contact condensation and evaporation for various flow regimes. Such "TOO EQUILIBRIUM" conditions have also been observed in RELAP-5/MOD-1 (Ref. 2). Further research on direct contact condensation at higher pressures may be needed to refine these physical models.

As for the cooldown discrepancy, an optimisation of the form loss factors in the recirculation loops in the steam generators may lead to better code results.

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FIG.6.1 VAPOR GENERATION RATE IN S.G.B. (DOME)



7. DISCUSSION OF THE RESULTS OF A SENSITIVITY STUDY

Table 7.1. (from Ref. 3) gives a summary of the main input-data modification between 3 simulation studies of the SGTR event. While preceding chapters were concentrated only on the results of case 3, this chapter will discuss those results of case 2 which may help to explain the observed anomalies explained in chapter 6.

7.1. Water level discrepancy in intact S.G. at start of cooldown

For case 2, the cooldown was initiated suddenly at 900 s, by instantaneous opening of the atmospheric steam dump valve of the intact S.G. (rather than a gradual opening during 300 seconds for the base case).

Fig. 7.1. illustrates a comparison between calculated and recorded water level in the intact S.G. for the 3 cases.

The sudden depressurisation produces a much larger level swell resulting in a level overshoot of 20 % (compared to about 8 % for the base case).

Fig. 7.1. also illustrates that the level swell phenomena occurred only for the cases 2 and 3 (RELAP-5 MOD-2) and not for case 1 (RELAP-5 MOD-1). A change in flow regime map between MOD-1 and MOD-2 could explain the basic difference.

Conclusion :

Too strong momentum coupling between the vapour and liquid phase leads to a wrong mass redistribution in the S.G., during a sudden depressurization.

7.2. Discussion of sensitivity study concerning the cooldown period

Fig. 7.2. illustrates the comparison between the heat transfer between primary and S.G.'s.

The basic differences between the nodalisation for case 3 (base case) and case 2 are related to the form loss coefficients introduced in the downcomer of the steam generators, and a slightly simpler nodalisation of the separator region as shown in fig. 7.3.

In the three downcomer junctions, the forward form loss factor was set at 200 (case 2) and zero (case 3 - base case), while the reverse form loss factor was set at 1000 for both cases 2 and 3.

A higher forward form loss factor for case 2 leads to a strong reduction in the natural circulation flowrate (factor 7) in the intact S.G. (at $t = 1600$ s) resulting in a lower cooling capacity as can be seen in fig. 7.2.

When the AF to the affected S.G. is stopped at $t = 1632$ s (tripped prematurely on high water level in intact S.G.), the reversed natural circulation in this S.G. is maintained and has a tendency to increase, in strong contrast to the base case when AF trips (at 1900 s).

It follows then that the cooldown rate of the RCS temperature is closer to the recorded value (fig. 7.4), and the depressurisation peak at the end of the spray period is also closer to the measured value (fig. 7.5.) for case 2.

Conclusion :

The natural circulation in the S.G. depends strongly on a proper choice of the form loss factors in the junctions along the natural circulation path, and has a very important impact on the primary RCS especially for asymmetric plant behaviour.

7.3. Vapour condensation phenomena in highly stratified conditions

In chapter 6.3., a vapour condensation anomaly was detected in the dome of the affected S.G. under following conditions :

- . very high temperature stratification
- . zero steam-and feed flow (bottled-up)
- . very high water level in the S.G.

Fig. 7.6. illustrates the S.G. pressure evolution in both S.G.'s for case 2 (parametric study). Concerning the affected S.G., the premature pressure rise at 1634 s, is caused by a trip of the turbine driven auxiliary feedwater pump at high S.G. water level in S.G A (Table 7.1), which also closes the steam flow to the turbine, and produces a repressurisation of the affected S.G.

Comparing the measured and the calculated pressure trend beyond 2400 s, one observes an acceptable pressurization rate, in contrast to the results of the base case (fig. 5.10.1) and discussed in chapter 6.3.

Table 7.2 below compares the main S.G. dome parameters for the base case and the sensitivity study (case 2) from a major edit at 2200 s.

The interpretation of the results should account for a different nodalisation between base case (fig. 4.1.) and parametric study (fig. 7.3.).

From this table several observation can be made.

1. For same the flow regime in the dome volume, (ie SLUG), the heat transfer regimes are very different.
2. The structural heat flux repartition between vapour phase and fluid phase is reversed between both cases. While for case 2, heat is exchanged between gas phase and outer structures, for case 3 (base case), heat is exchanged only between water phase and outer structure.
3. For case 2, heat losses from the vapour phase to the structures produces the vapour condensation on the walls and run - off in the water phase which corresponds to the vapour condensation flux calculated.
4. For case 3, the large vapour condensation results only from a direct contact condensation between the vapour and the

water phase, since no heat is extracted from the vapour phase by the colder structures.

Conclusion :

- a/ The absence of the large direct contact condensation in the sensitivity study results in a heat balance which yields an acceptable pressure trend.
- b/ The selection criteria for the heat transfer regime for condensation in RELAP should be reviewed in the light of above observation.

TABLE 7.1

TABLE 1 : SUMMARY OF SGTR - ACCIDENT SIMULATION NODALISATION AND RESULTS

CASES	1	2	3
date of execution	27 june 83	19 april 85	28 september 85
code : RELAP-5	MOD 1 CYCLE 014	MOD 2 CYCLE 36	MOD 2 CYCLE 36.01
separator model	MOD 1	MOD 2 VOVER = 0.1 VUNDER = 0.1	MOD 2 VOVER = 0.1 VUNDER = 0.1
SG plenum + bypass	-	no	yes
break area (cm ²) (VCMIS)	1.015 (0000)	1.724 (00003)	1.724 (00003)
Forward loss coefficient in SG downcomer	3 X 200	3 x 200	-
letdown closure, back-up heaters off	t = 211 s	$L_{PR} \leq 20\%$	$L_{PR} \leq 20\%$
AFW (turbopump) off	t = 1900 s	LSG $\geq 45\%$	t = 1900 s
spray nozzle location	TOP (TDJ) (EXTERNAL)	TOP (TDV + TDJ)	AS BUILT (TDJ)
SGA cooldown initiated	t = 1000 s	t = 1000 s	$900 \leq t \leq 1200$
structural heat losses to containment	NO	NO	YES
Number of volumes	136	144	154
Number of junctions	140	150	150
Number of heat slabs (mesh points)	157 (463)	161 (553)	164 (563)
TIME - STEP LIMITATION (PRINCIPAL)	MASS ERROR	TRANSIT TIME (HOT LEG)	TRANSIT TIME (U LEG)
ACCUMULATED MASS ERROR RATIO	$- 79 \cdot 10^{-4}$	$\sim 1.14 \cdot 10^{-4}$	$1.08 \cdot 10^{-6}$
AVERAGE TIME STEP (s)	0.09	0.124	0.124
CPU - TIME / TRANSIENT-TIME	22.6	18.0	19.4

TABLE 7.2.

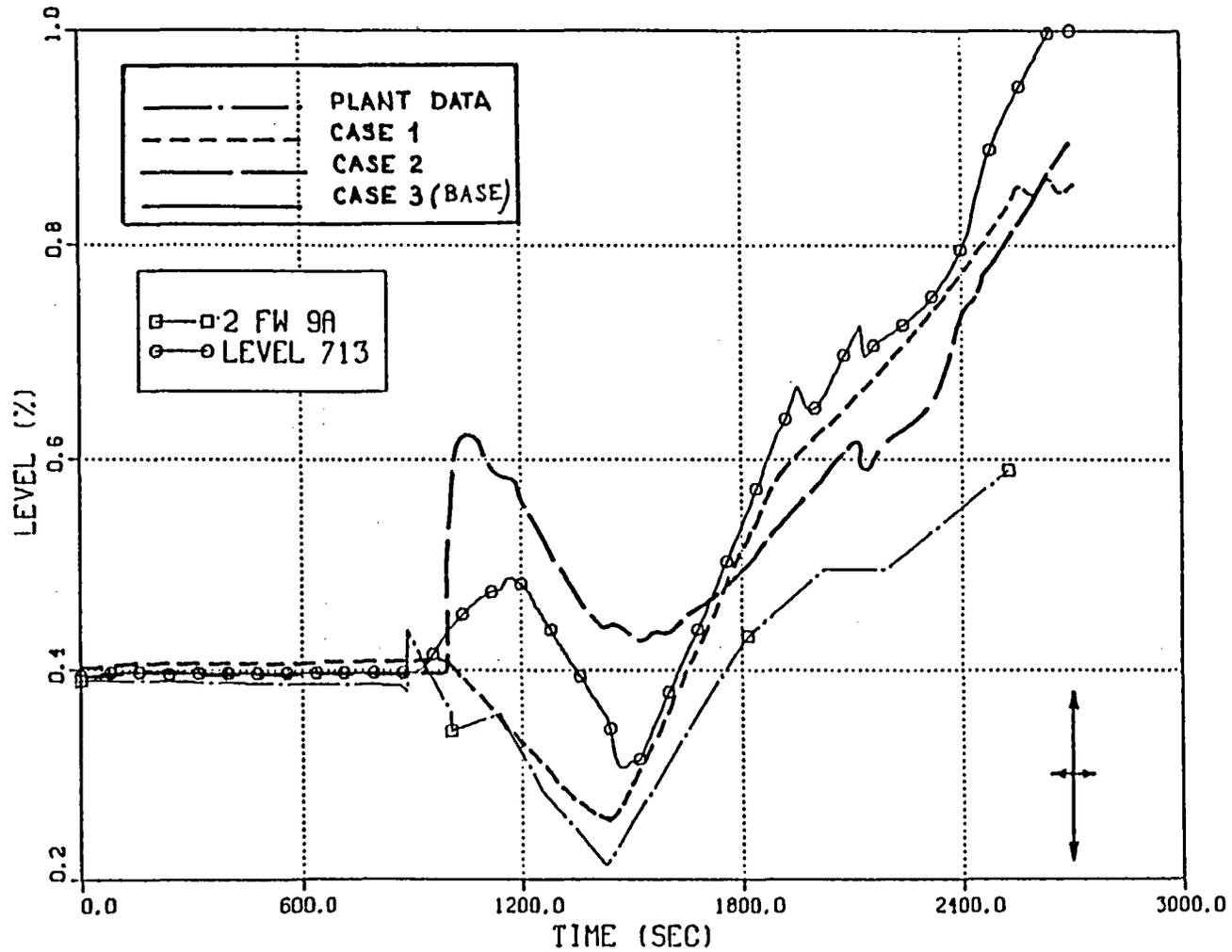
COMPARISON TABLE OF S.G. DOME VOLUME DATA AT 2200 S

PARAMETER	UNITS	SENSITIVITY STUDY	BASE CASE
Case	-	2	3
Height of dome component	m	3.093	2.093
Volume of dome component	m ³	38.85	26.29
Dome void	fraction	0.652	0.704
Dome flow regime	-	SLUG	SLUG
Dome saturation temperature	°K	540	517
Dome gas temperature	°K	552	523
Dome fluid temperature	°K	515	507
Dome outer heat slab surface temperature	°K	540	517
Heat transfer regime	-	condensation (11)	Subcooled nucleate boiling (3)
Heat fluxes			
a. Internal struct. to gas	watt	644	29 903
b. External struct. to gas	watt	- 76 320	0
c. External struct. to fluid	watt	0	163 442
c. Total flux to volume	watt	- 75 676	193 346
d. Tot. flux to gas	watt	- 78 502	- 406 216
Vapour generation rate	kg/m ³ , s	- 0.00124	- 0.00824

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FIG. 7.1 MEASURED WATER LEVEL IN STEAM GENERATOR A



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DOEL2 SGR INCIDENT 25 JUNE 1979 85/09/28.
FIG 7.2 HEAT TRANSFER RATE FROM PRIMARY TO SG

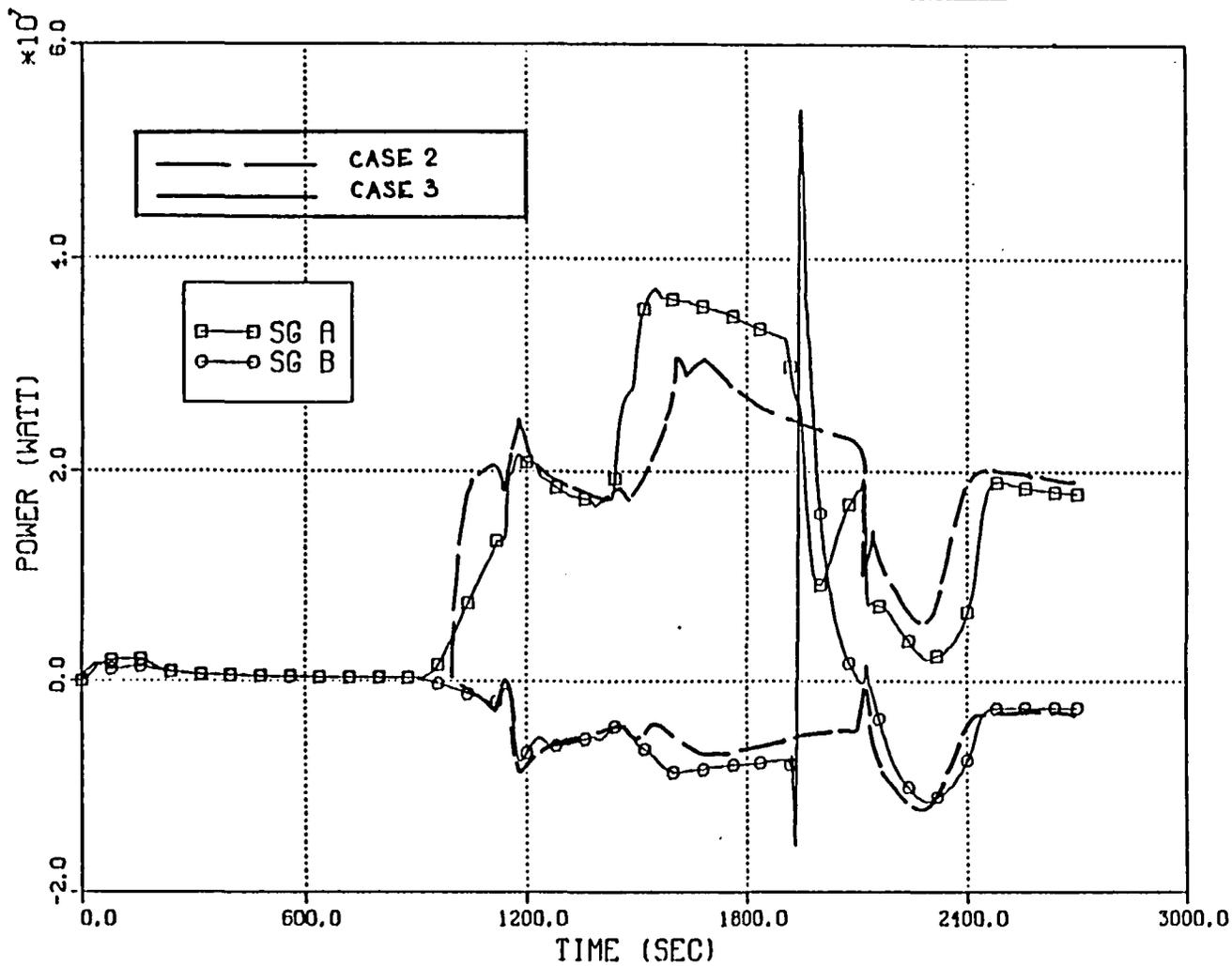
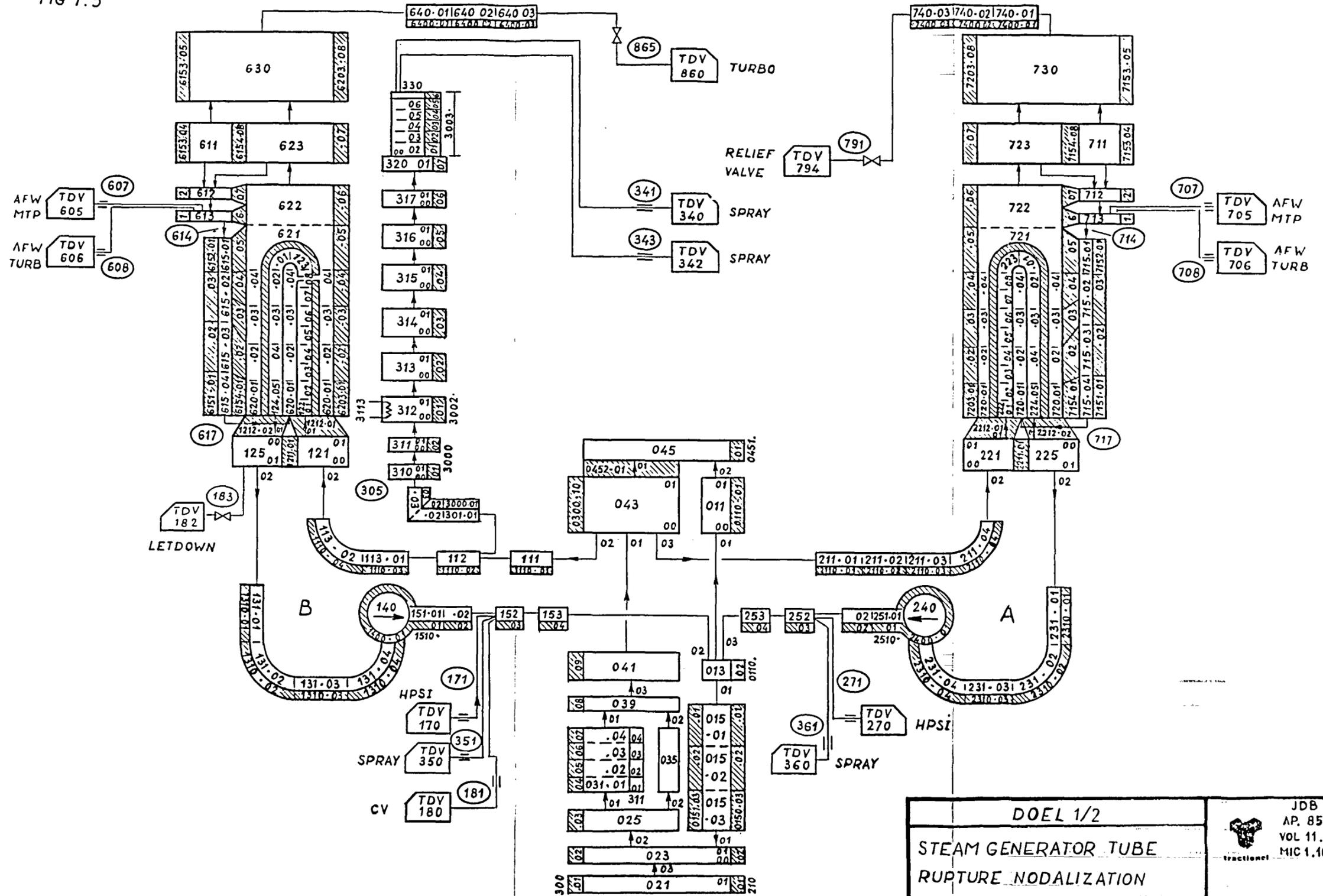


FIG 7.3

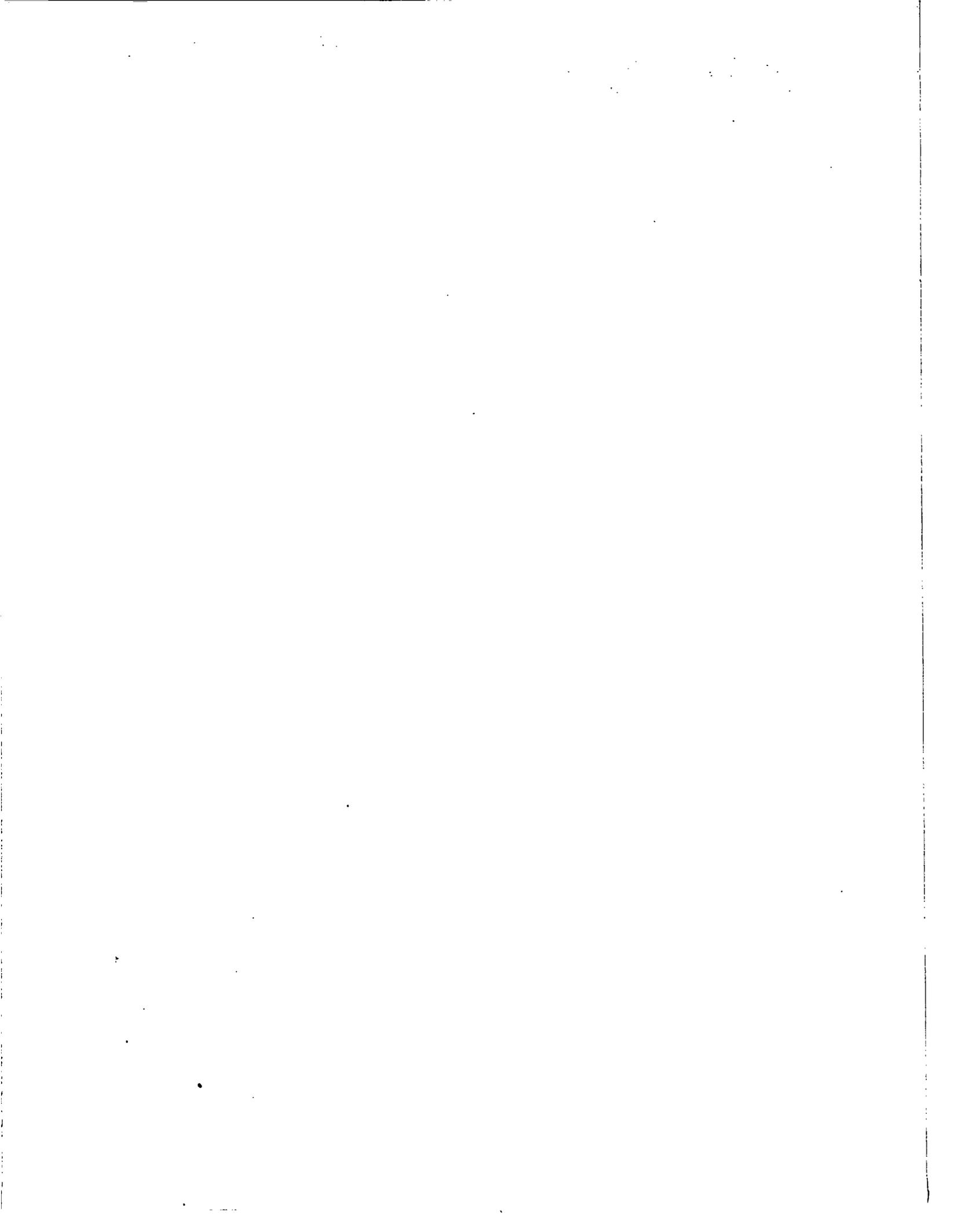


DOEL 1/2

STEAM GENERATOR TUBE RUPTURE NODALIZATION

JDB AP. 85 VOL 11.6 MIC 1.10

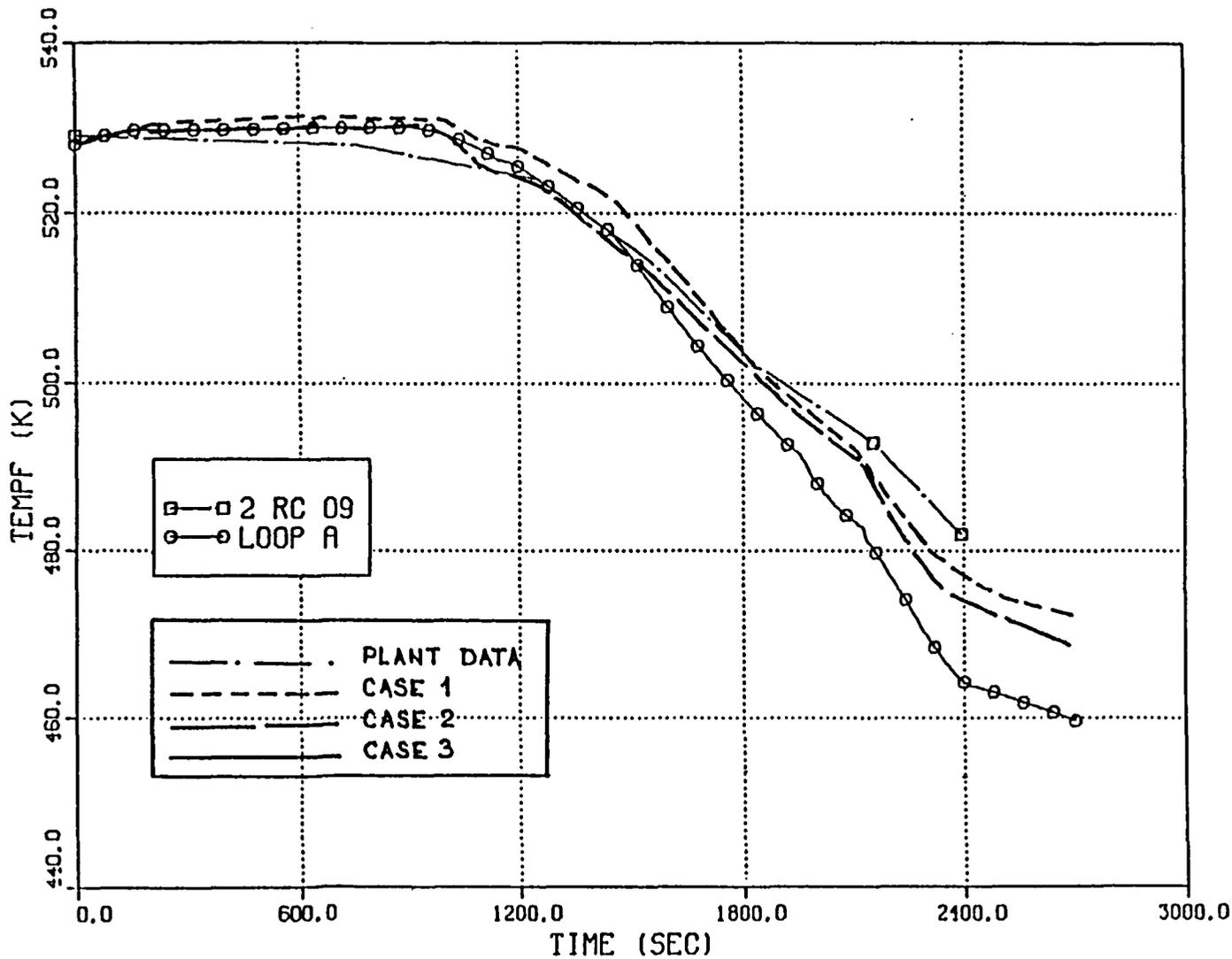
AFDRUK VOORBEHOUDEN ZONDER ONZE TOESTEMMING MAG NIET PLAK NIET GEKOPIEERD NIET VERVEELVULD WORDEN NC



RELAP5/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM

DOEL2 SGTR INCIDENT 25 JUNE 1979 05/09/28..

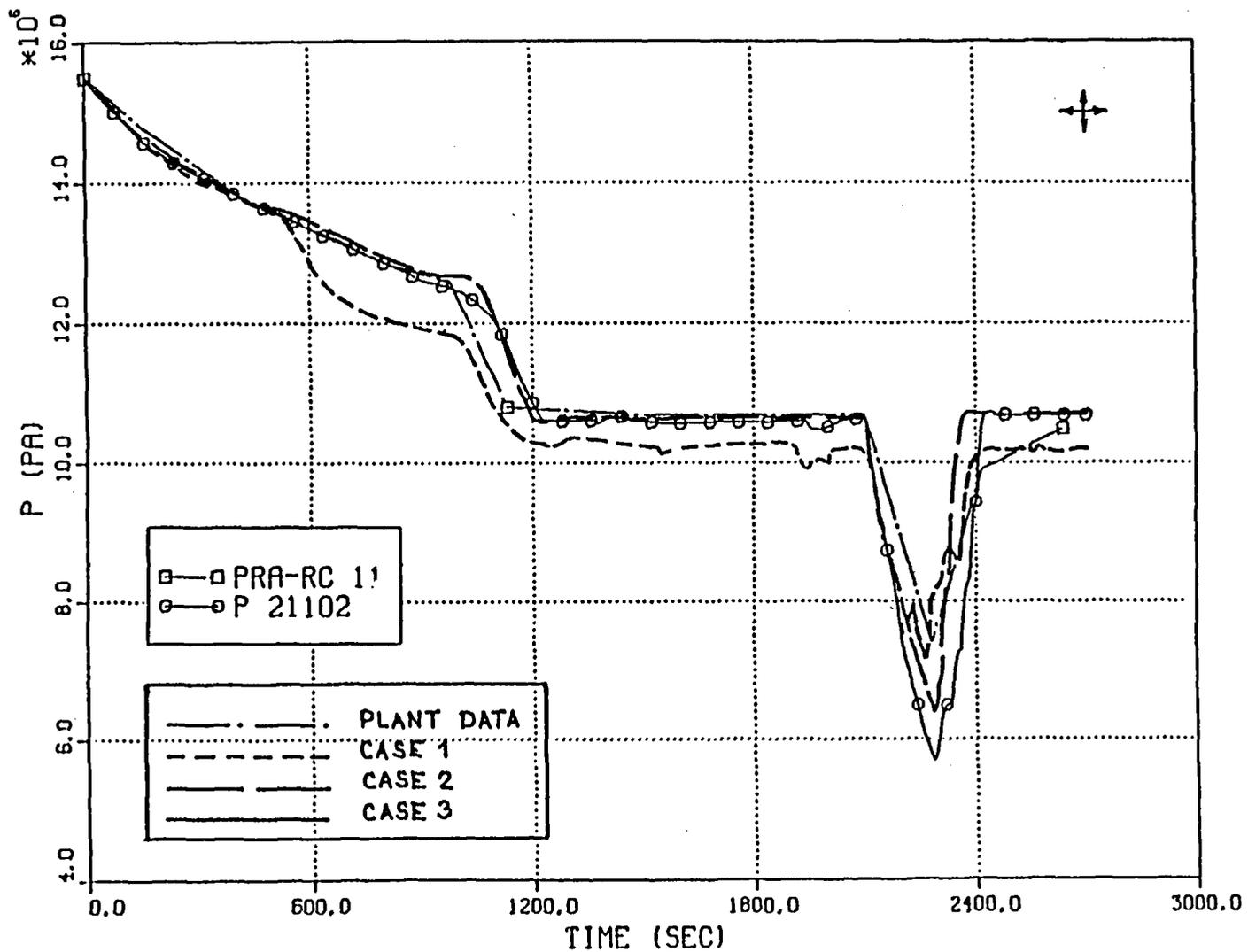
FIG.7.4 TEMPERATURE EVOLUTION IN COLD LEG A



RELAP5/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM

DOEL2 SGTR INCIDENT 25 JUNE 1979 85/09/28.

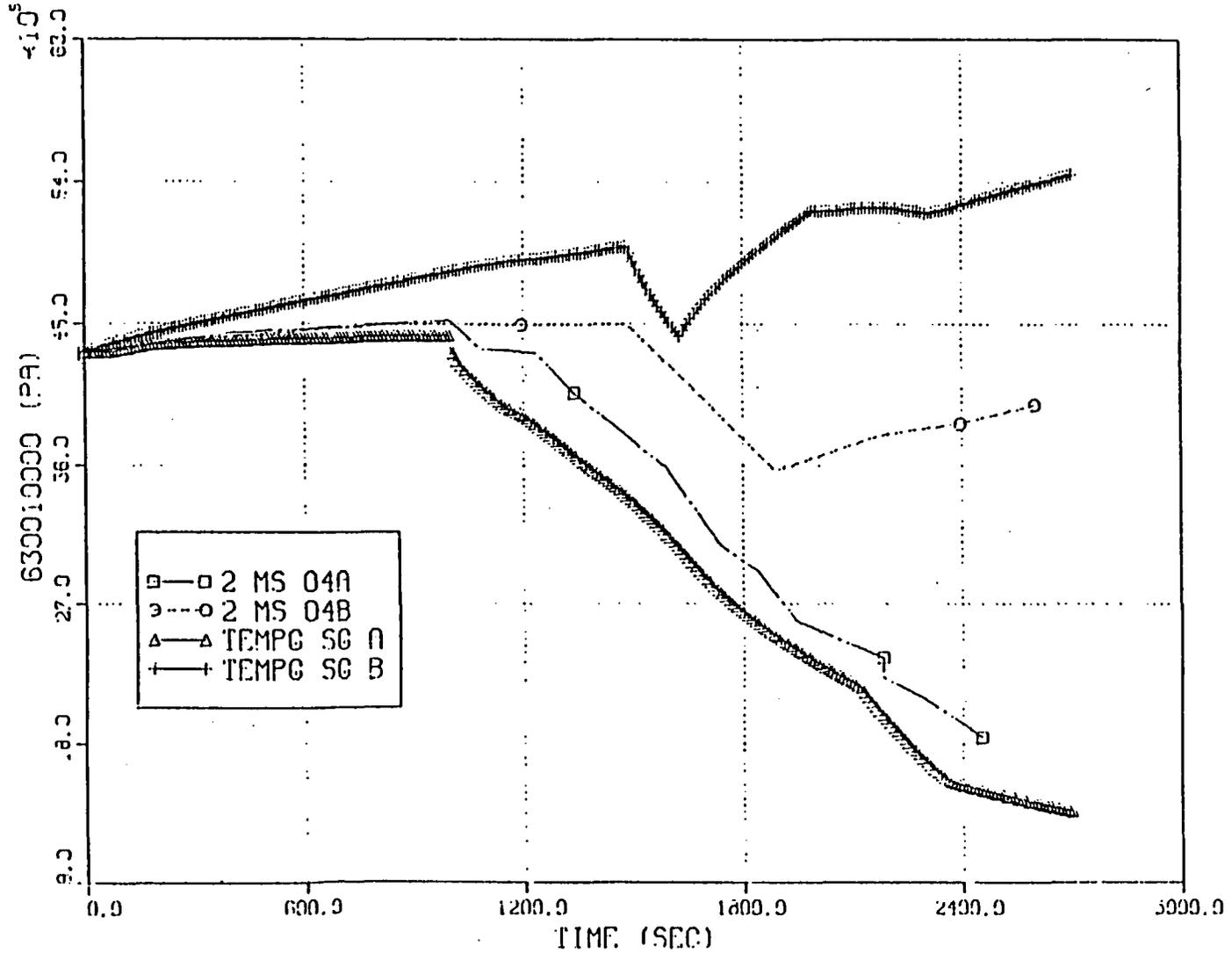
FIG 7.5 PRESSURE EVOLUTION IN HOT LEG A



RELAP5/MOD2/03G REACTOR LOSS OF COOLANT ANALYSIS PROGRAM

SGTR BENCHMARK 85/04/19.

FIG 7.6 PRESSURE EVOLUTION IN STEAM GENERATORS



8. RUN STATISTICS

In order to compare run times it should be mentioned that the CPU time for the CYBER 176 machine of the CDC CYBERNET system does not correspond with a clock time. For billing purposes, the CPU time has been manipulated such that for all machines of the service center, the same equivalent CPU is applicable. Hence, it turns out that for the CYBER 176 machine the clock time is about 11 times smaller than the printed CPU time.

In further discussions, clock time will be used (as distinct from CPU-time) in order to arrive at run statistics comparable with statistics of other ICAP reports.

Fig. 8.1. illustrates the CPU-time evolution versus transient time.

The requested time step was constant at 0.25 s.

The average time step varies between 0.122 and 0.125 s.

The time step limitation was "minimum Courant" for most of the time steps. The limitation occurred systematically in volume 13302 (vertical pump suction segment of intermediate leg) and 25102 (segment in cold leg between pump and SI injection):

volume	length	max. velocity	max. Courant	possible gain
(-)	(m)	(m/s)	s	%
13302	1.43	9.861	0.145	14
25101	1.968	14.	0.14	12

Table 8.1. illustrates various run statistics at various transient times.

From this table one observes that the maximum mass error goes from + 10.5 kg to - 10.0 kg or a max. fraction of $\pm 3.5 \cdot 10^{-6}$

Fig 8.1 shows that the CPU TIME versus transient time is very constant and averages out to 19.3 CPU-s/transient-s or 1.76 clock-s/transient-s.

The performance number

$$P.R. = \frac{1\ 000 \times \text{clock-s} \quad 1\ 000 \times \frac{52\ 249 - 122}{11}}{\text{vol} \times \text{DT} \quad 154 \times 21\ 883} = 1.406$$

Conclusion :

In contrast to previous calculations by means of the Code RELAP-5 MOD-1, the accumulated mass error remains low, and the running time is very stable, with a performance ratio of about 1.8 (execution/transient time)

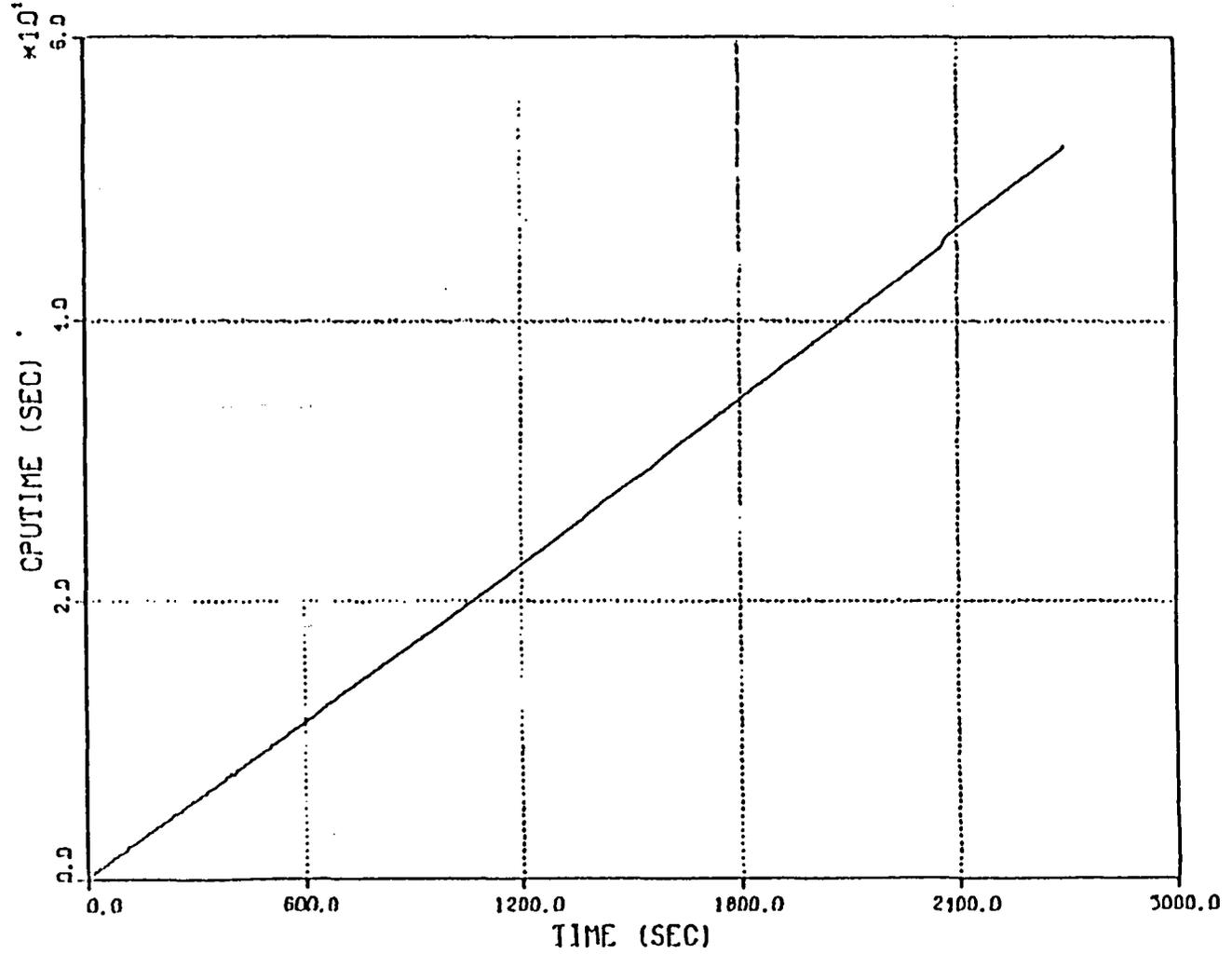
TABLE 8.1 - RUN STATISTICS TABLE

TRANSIANT TIME	CPU- TIME	CLOCK TIME	TOTAL MASS KG	MASS ERROR FRACTION	AVERAGE ΔT	NUMBER OF CYCLES	REPEATED ADV
0	122	11.1	0	0.		0	
200	3855	350	4.46	1.5 E-6	0.1249	1602	-1
400	7546	686	6.53	2.15 E-6	0.125	3202	-1
600	11250	1023	8.37	2.75 E-6	0.125	4802	-1
800	14930	1357	9.66	3.17 E-6	0.125	6402	-1
1000	18666	1697	6.92	2.27 E-6	0.125	8002	-1
1200	22565	2051	1.63	0.54 E-6	0.125	9602	-1
1400	26550	2414	10.503	3.45 E-6	0.124	11232	-12
1600	30564	2779	3.06	2.80 E-6	0.122	12891	-40
1800	34413	3128	7.11	2.31 E-6	0.125	14491	-40
2000	38363	3488	-0.025	0.008E-6	0.124	16110	-48
2200	42210	3837	-10.09	-3.25 E-6	0.1244	17723	-54
2400	46561	4233	-3.649	-1.17 E-6	0.115	19479	-72
2600	50347	4577	1.663	0.531E-6	0.1248	21083	-74
2700	52249	4750	3.39	1.08 E-6	0.125	21883	-74

RELAP5/2/36.01 REACTOR LOSS OF COOLANT ANALYSIS PROGRAM

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FIG 8.1 CPUTIME VERSUS TRANSIENT TIME



9. CONCLUSIONS

An assessment study for the code RELAP-5 MOD-2 CYCLE 36.01 was presented in this report based on a real plant transient (DOEL-2 SGTR event).

Although the quality and quantity of the plant data in terms of plant diagnostics and recordings may be considered poor with respect the known thermal-hydraulic facilities, it was shown that comparison of parameter trends can lead to an opinion on the strength and weaknesses of the code to simulate the observed phenomena.

Such as :

1. The code is capable to simulate the observed phenomena reasonably well and can be used to predict similar transients. However, lack of precise boundary conditions does not allow one to quantify the error margin from this analysis.
2. The code performance is acceptable. The time step is constant and Courant limited for 99 % of the transient, and the execution time on a CYBER-176 is about 1.8 times the transient time. The mass error ratio remains negligible.
3. Impressive improvements over RELAP-5 MOD-1 CYCLE 19 have been observed in the steady break flow rate, and the water-vapour flow slip (CCFL) in the pressuriser, when pressuriser spray is actuated.
4. Excessive water level swell observed in the intact S.G. during cooldown may be attributed to excessive interphase momentum transfer in the S.G. riser region when bulk boiling is initiated.
5. Excessive interphase mass and heat transfer for condensation and evaporation in quasi stagnant flow conditions was observed in the vapour space of the isolated affected steam generator.

6. Erratic natural circulation in the affected steam generator leads to an anomalous cooldown capacity of the steam generator. This behaviour can be suppressed by inserting high form loss factors in the natural circulation flow path.

7. While some anomalies may be reduced by a judicious choice of the nodalisation and/or code options, this study suggests that further improvements in the constitutive equation package, in function of the various flow regimes is needed, to take full benefit of the potential of a 6 equation code.

10. REFERENCES

1. Analysis of Steam Generator Tube Rupture Events at Oconnee and Ginna. INPO - 82-030 November 1982.
2. Analysis and simulation of the DOEL-2 SGTR event. E. J. STUBBE et al.
ANS PROCEEDINGS : Anticipated and Abnormal plant transients in LWR's, Vol 1 pp 183-192, September 1983.
3. Application of RELAP-5 to the analysis of the Doel-2 SGTR accident, and studies of the Doel 1-2 loss of feedwater and feedwaterline break accidents.
E. J. STUBBE, L. VANHOENACKER
Presented at the 13th Water Reactor Safety Research information meeting, GAITHERSPURG Oct. 22-25 1985.
4. RELAP-5/MOD2 CODE MANUAL
V. H. RANSOM et al, NUREG/CR-4312, EGG-2396, August 1985

REPORT ON THE INCIDENT AT DOEL 2 NUCLEAR POWER PLANTSEVERE LEAKAGE IN STEAM GENERATOR B ON JUNE 25, 19791. Status of the power plant at the moment of the incident

The primary system was being heated up after repair works at the actuation system of the main steam valve.

At the moment of the incident, temperature in the primary system was $\pm 255^{\circ}\text{C}$ (point A on Fig. 1 & 2) and pressure had reached its rated value of 157 kg/cm^2 (point A on Fig. 3 & 4).

The reactor was subcritical with all rods in.

Secondary pressure in the steam generators was $\pm 45 \text{ kg/cm}^2$, the saturation pressure corresponding to 255°C (point A on Fig. 6 & 7).

For some time, A-loop steam generator had shown a low

A/ activity value along the secondary side.

2. Sequence of the events (refer also to various computer data given in attachment)2.1. Initiating phase

A/ About 7:20 PM, a quick level decrease (Fig. 5) and a pressure decrease (about 2 kg/cm^2 per minute : see Fig. 4) are recorded in the pressurizer. The level decrease results in an acceleration of the operating charging pump. A second

A/ charging pump is started manually. It is confirmed that the relief valves are closed, and their isolation valves are preventively closed. The level in the pressurizer continues to decrease, and as a result the letdown station of the CV system closes. The electrical heaters are disconnected, which is confirmed manually.

The steam bubble expands into the primary loop.

At the same time, a quick level increase is recorded in B-loop steam generator (point B on Fig. 7).

A/ The importance of the leak has been later evaluated at 40 t/hr. The activity measurement channels of the blowdown system record a maximum value.

The combination of all those signals indicates a severe leak in B-loop steam generator. The faulted steam generator is then immediately completely isolated along the steam side,

A/ but the line to the turbo-pump is omitted.

The discharge valve to the atmosphere is set at maximum

A/ pressure (80 kg/cm²).

Meanwhile, the third charging pump is started (was set apart to be maintained), but three charging pumps are not sufficient to compensate the loss of fluid in the steam generator.

Indeed, the CV tank is readily empty, and the charging pumps are automatically supplied from the pool fill tank.

A/ Cooling of the primary system starts through a discharge to the atmosphere of A-loop steam generator, in which pressure and temperature quickly decrease (point A on Fig. 6).

Primary pump B is stopped.

2.2. Actuation of the safety injection

About 20' after the incident started, the threshold pressure (118.5 kg/cm²) to actuate the safety injection is reached. The emergency diesels start within the required time lapse but are not necessary. Phase A isolation and ventilation isolation of the reactor building are achieved. The vital components not yet in operation are started.

When reaching the 108 kg/cm² value, all HP SI-pumps discharge into the primary system, and the pressure decrease is stopped (point C on Fig. 3).

A/
As the low level threshold is exceeded in A-loop steam generator, the auxiliary feed pump starts. The discharge through this turbo-pump results in a quick pressure decrease in B-loop steam generator (point C on Fig. 7) for about 8 minutes, time during which ± 1 ton steam could be discharged (until a level of about 45% in the two steam generators). To decrease primary pressure and thus also the leak rate, spray is maximum in the pressurizer. Primary pump B is started, and both spray lines are used (point D on Fig. 3). During this phase, the steam bubble in the pressurizer completely disappears, and it fills up with primary water at $\pm 230^{\circ}\text{C}$. Spray is stopped at ± 75 kg/cm², and pressure stabilizes at a value corresponding to the head of the four SI-pumps taking into account the leak flow.

The auxiliary feed pump of the faulted steam generator is stopped locally and isolated (point D on Fig. 7). This cannot be performed from the control room since the SI signal still prevails. The auxiliary feedwater supply tank is filled up from Doel 1.

2.3. Cancelling of SI-signal

Pressure decrease was now mandatory :

- (a) to avoid the preset opening of safety valves of the faulted steam generator
- (b) to start, as soon as possible, the shutdown cooling system (low pressure circuit !) to stop the letdown of slightly contaminated steam through the A-loop steam generator.

First, the safety injection signal had to be cancelled.

A/ This had to be performed more than once (each time requiring 5 minutes interval) because the relay did not start within the required time lapse. Finally, the concerned bistables had to be flicked over.

After definitively cancelling the SI-signal, two HP SI-pumps are stopped and soon thereafter a third one. While considering the subcooling margin, the last HP SI-pump is stopped, and pressure successively decreases to reach $\pm 65 \text{ kg/cm}^2$ (point H on Fig. 3).

A/ At that moment, the three charging pumps are running. It is then tried to initiate the CV-discharge line, but valves do not open. Some time goes by before the reason therefore is determined. Due to phase A isolation, there is no longer a compressed-air supply in the reactor building. After re-opening the compressed-air supply line, the discharge line is opened (point I on Fig. 3).

A/ A charging pump is stopped, pressure decreases, first quickly, then slower.

A/ The loss of compressed-air supply has also resulted in the closure of the CC-valves to the thermal shield of the primary pumps. This lasts for about 10 minutes, but no damage was recorded.

2.4. Initiation of the residual heat removal system

As the CV-system permitted only a slow pressure decrease, the interlock, which maintains the isolation of the RHRS up to a pressure of 28 kg/cm^2 , has been bypassed at 31 kg/cm^2 . There was indeed a sufficient margin compared to the design pressure of the system (42 kg/cm^2). Thanks to this operation, the letdown through A-loop steam generator could be stopped earlier and the discharge of slightly contaminated steam could be reduced (point J on Fig. 3).

A// The signal ordering the automatic isolation of the system is again operational. However, shortly thereafter isolation starts again (pressure peak at 37 bar) due to a misregulation of pressure. Isolation is again bypassed to allow starting the RHRS.

2.5. Further sequences

Primary pressure decreases to become lower than the secondary pressure in B-loop steam generator. Secondary level decreases, which may create a dilution risk. The boric acid concentration is controlled every half hour (stabilized at $\pm 1500 \text{ ppm}$).

Thanks to cooling down, pressure decreases slowly in B-loop steam generator and reaches a value lower than the primary pressure. From this moment on, attention is paid to always maintain primary pressure above that in the steam generator. Despite the cold water so discharged in the steam generator, pressure decreases only slowly (due to the presence of a warm water film at the water surface).

As the level of water in the steam generator approaches the

upper limit of the broad level measurement, pressure is sufficiently low ($\pm 12 \text{ kg/cm}^2$) to allow nitrogen injection. The secondary drain line is coupled with system B for liquid waste, and the steam generator discharges into it through the nitrogen pressure.

As the nitrogen is only slightly contaminated, it is possible to discharge it to the atmosphere via the annulus between primary and secondary containments.

A/ 2.6. Comments and conclusion

- 1) The incident has been mastered as prescribed and has affected neither the environment nor the installation.
- 2) The procedure has to be reviewed with regard to the cancellation of phase A isolation to restore compressed air supply to the reactor building.
- 3) It has to be checked if some changes are not required in the IA system within the reactor building.
- 4) The further analysis of the incident has been hard to perform due to the poor quality of the computer data.

ATTACHMENT 1 - COMPUTER DATA1. Initiating phase

19 21'06" : pressurizer pressure below reference pressure
 19 22'51" : demand for accelerating charging pump
 19 23'31" : pressurizer heaters disconnected due to
 low level
 19 23'32" : CV letdown station valves closed
 19 25'42" : isolation valves of relief and spray valves closed
 19 26'14" : low pressure in primary system
 19 30'30" : very low pressure in pressurizer
 19 30'30" : high level in B-loop steam generator
 19 38'32" : B primary pump disconnected

2. Safety injection phase

19 40'18" : low pressure in pressurizer
 19 40'19" : safety injection due to pressurizer low pressure
 19 40'19" : diesels started
 19 40'19" : isolation of reactor building ventilation
 19 40'20" : phase A isolation of reactor building
 19 40'24" : actuation signal of HP SI-pumps
 19 40'33" : HP SI-valves opened
 19 43'48" : very large auxiliary feedwater flow to A SG
 A/ 19 43'54" : low level in A SG
 19 44'39" : very large auxiliary feedwater flow to B SG
 A/ 19 44'42" : valve from B SG to turbo-pump opened
 A/ 19 44'51" : valve from A SG to turbo-pump opened
 19 53'12" : auxiliary feed pump B disconnected
 19 56'37" : very low level in auxiliary feedwater tank
 19 57'11" : pressurizer level back to normal
 19 57'29" : pressurizer heaters re-started
 19 58'48" : high level in pressurizer

3. SI-signal cancelling phase

- 20 00'15" : diesel automatic starting signal cancelled and
SI-pumps actuation signal cancelled
- 20 00'21" : back to SI
- 20 03'24" : LP compressed air in the reactor building
- 20 05'59" : safety injection cancelled
- 20 06'05" : safety injection
- 20 10'59" : reactor building ventilation isolation cancelled
- 20 21'15" : HP SI-pump B disconnected
- A/ 20 22'36" : accumulator A valve closed
- 20 25'22" : HP SI-pump A disconnected
- 20 38'33" : valve CC-096 closed
- 20 40'25" : valve CC-099 closed
- 20 48'54" : compressed air supply to reactor building restored
- 20 49'00" : primary pumps CC valves re-opened

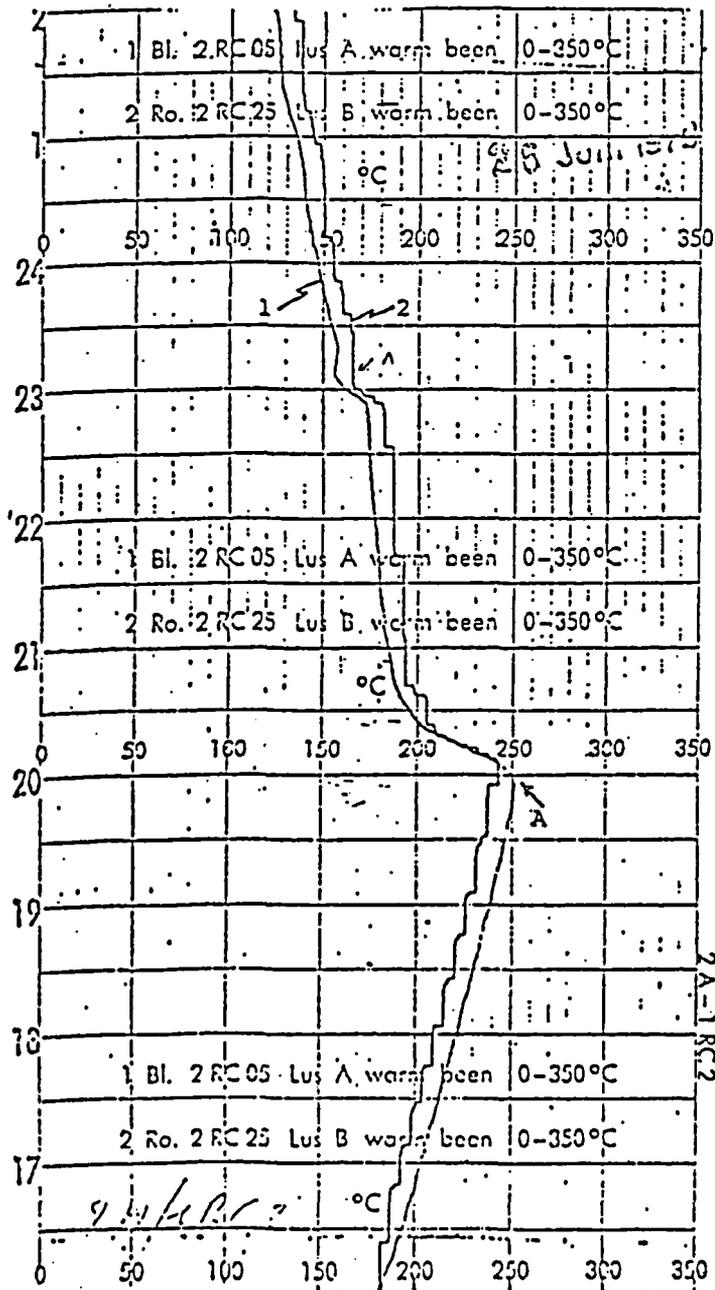
4. Actuation of RHRS

- 22 35'54" : valves RC-003 opened

Temperature hot leg
TEMPERATUUR WARM BEEN

FIGUUR 1

- Schrijver : 2 A - 1 RC 2
loop A - hot leg
 1. Bl. RC 05 lus A warm been 0 - 350°C
loop B hot leg
 2. Ro. RC 25 lus B warm been 0 - 350°C



Temperature cold leg

TEMPERATUUR KOUD BEEN

FIGUUR 2

Schrijver : 2. A - 1 RC 1

loop A cold leg

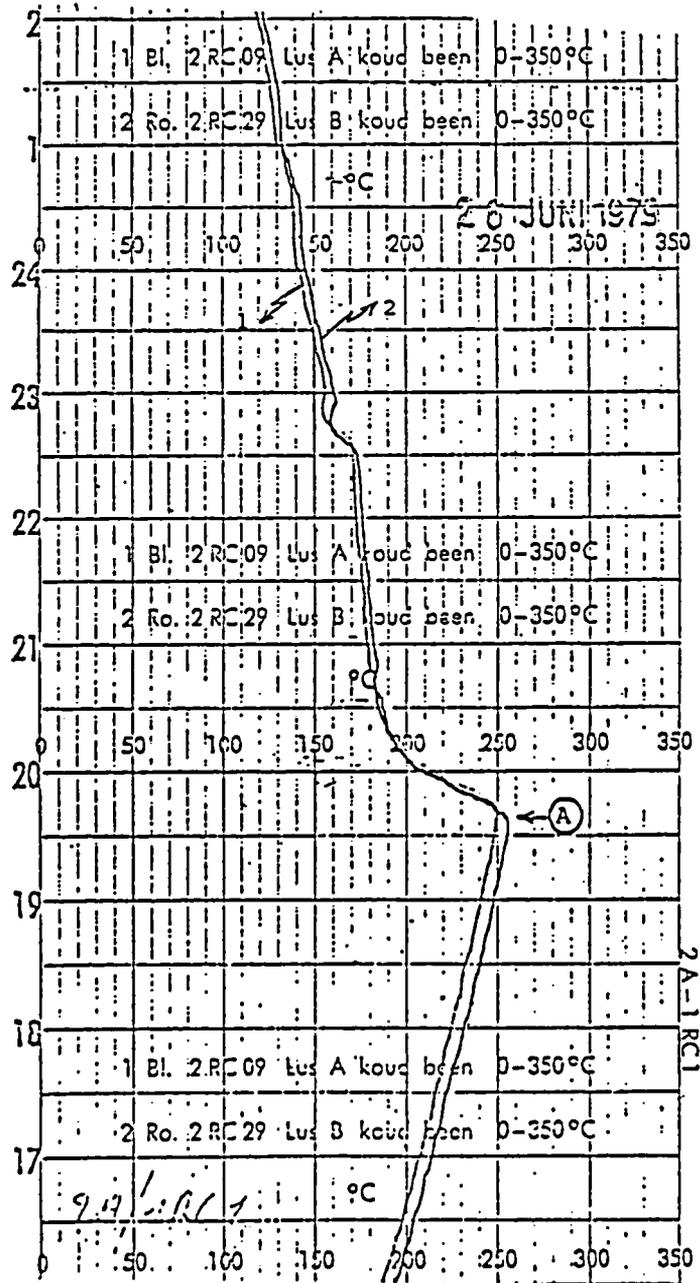
1. Bl. RC 09 lus A koud been

0 - 350°C

loop B cold leg

2. Ro. RC 29 lus B koud been

0 - 350°C

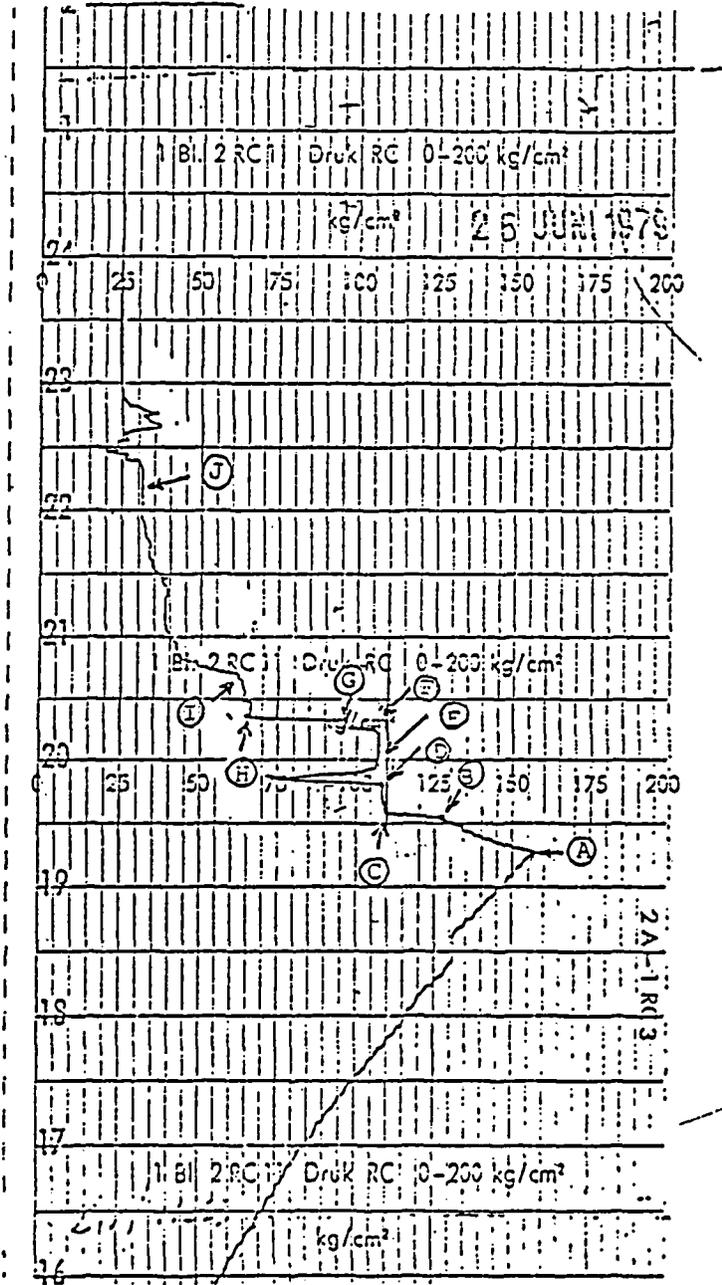


Presur RC
DRUK RC

FIGUUR 3

Schrijver : 2 A - 1 RC 3

1. Bl. RC 11 druk RC *Presur RC* 0 - 200 kg/cm²

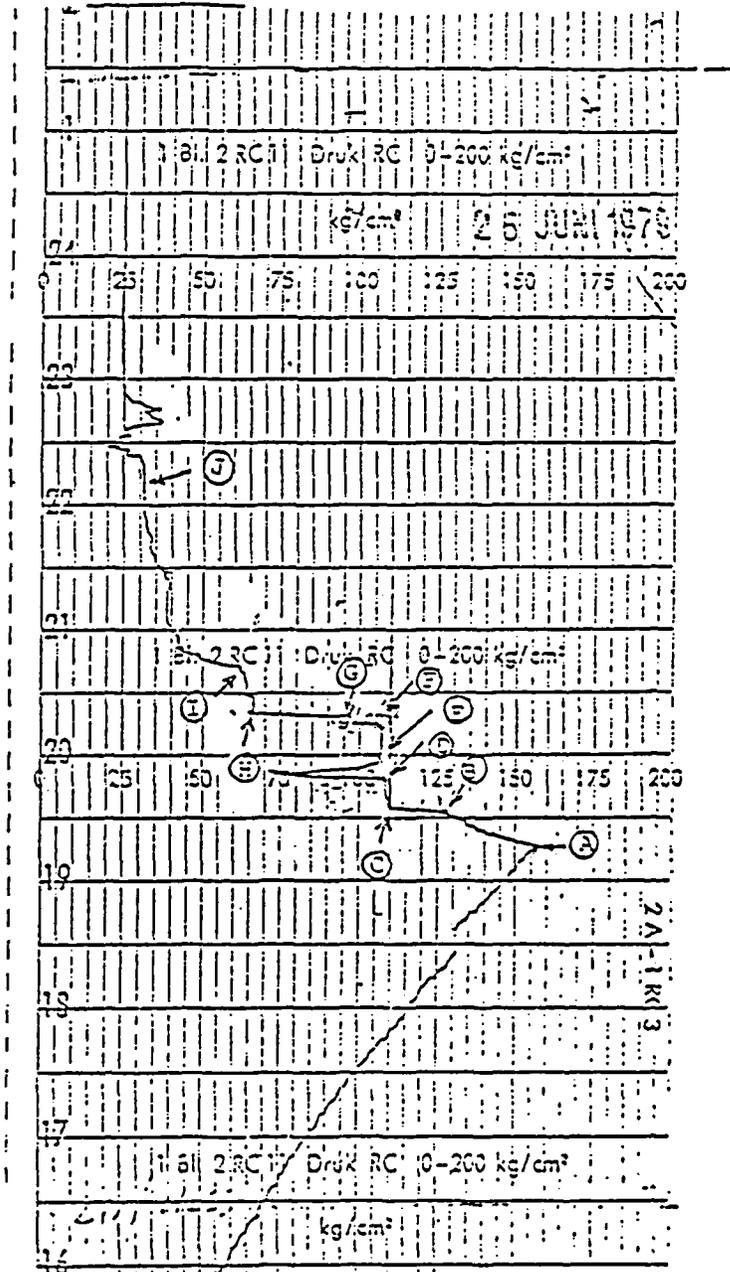


Pressure RC
Druk RC

FIGURE 3

Schrijver : 2 A - 1 RC 3

i. Bl. RC ii druk RC *Pressure RC* 0 - 200 kg/cm²



Pressing Pressure

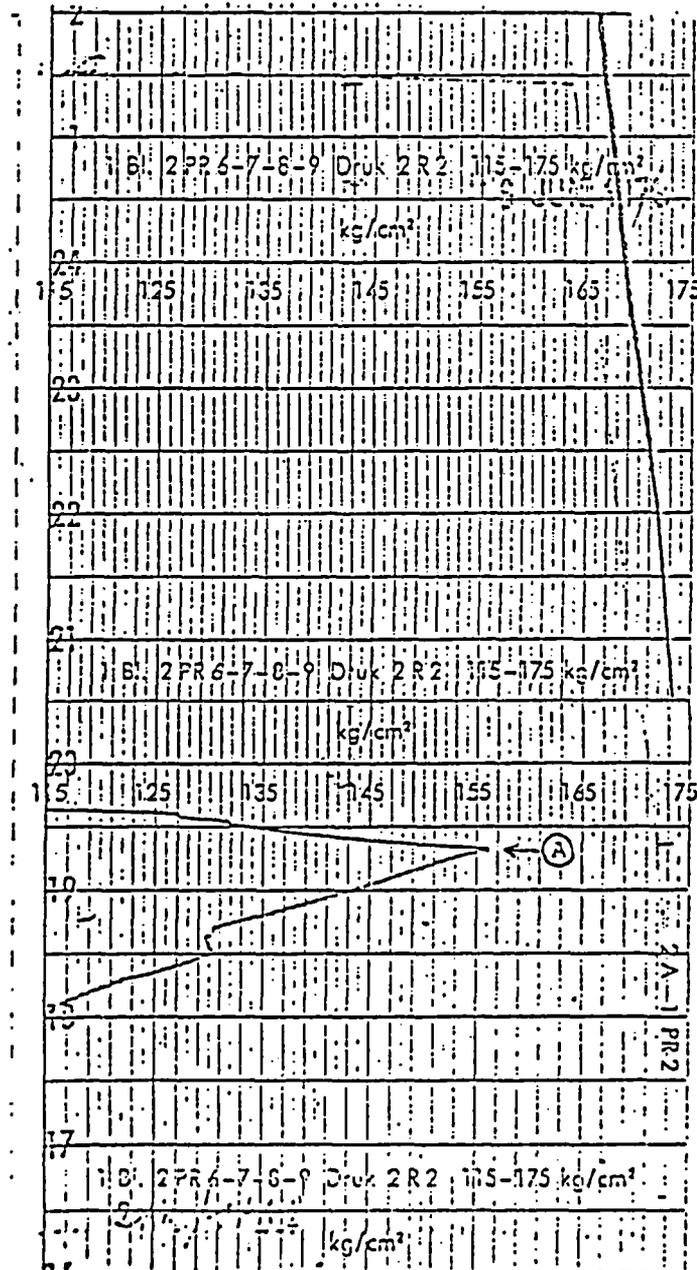
DRUK R 2

FIGUUR 4

Schrijver : Z. A - 1 PR 2

1. Bl. PR 6 - 7 - 8 - 9 Druk R2 *Pressing pressure* 115 - 175 kg/cm²

Meting in dienst spoor 1 :



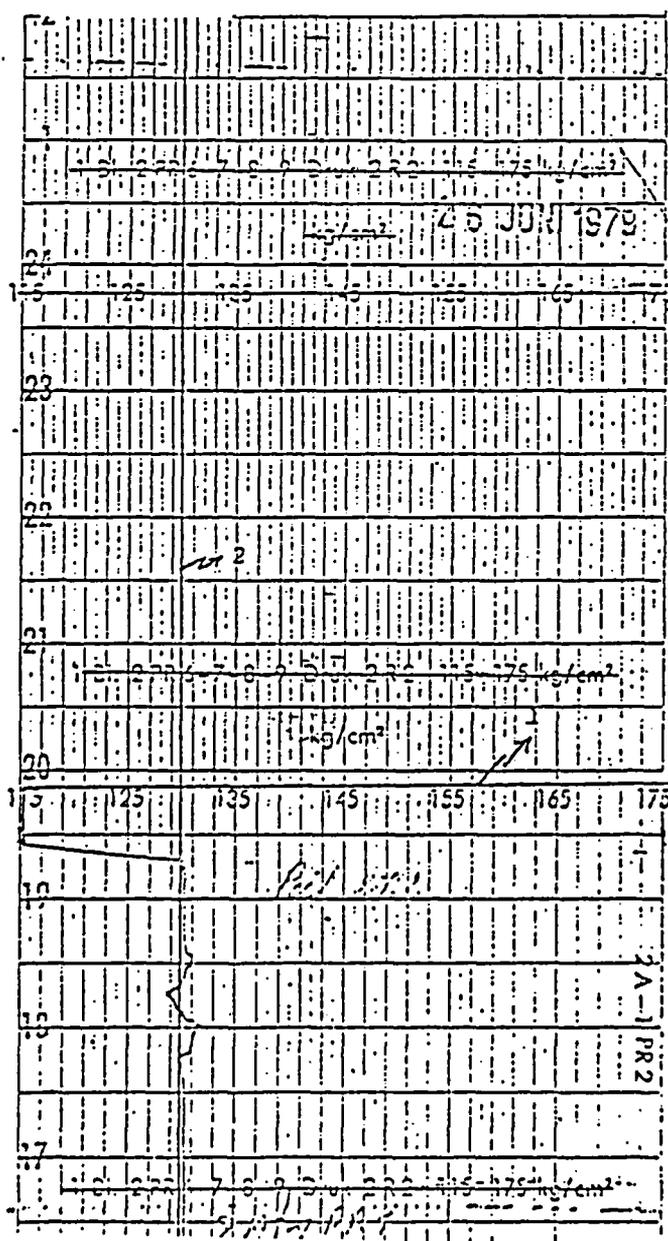
Pressurings level
PEIL R 2 FIGUUR 5

Schrijver : 2 A - 1 PR 1 *Pressurings level*

1. Bl. L. PR 11 - 12 - 13 Peil R 2 0 - 100 %

2. Ro. L. Ref. *Reference level*
 Ref. Peil R 2 0 - 100 %

Meting in dienst spoor 1 :



verkeerde schaal
 vervangen door
 0 + 100 %

wrong scale
 Replaced by
 0 - 100 %.

SG A - level and Pressure

SG. A PEIL - DRUK

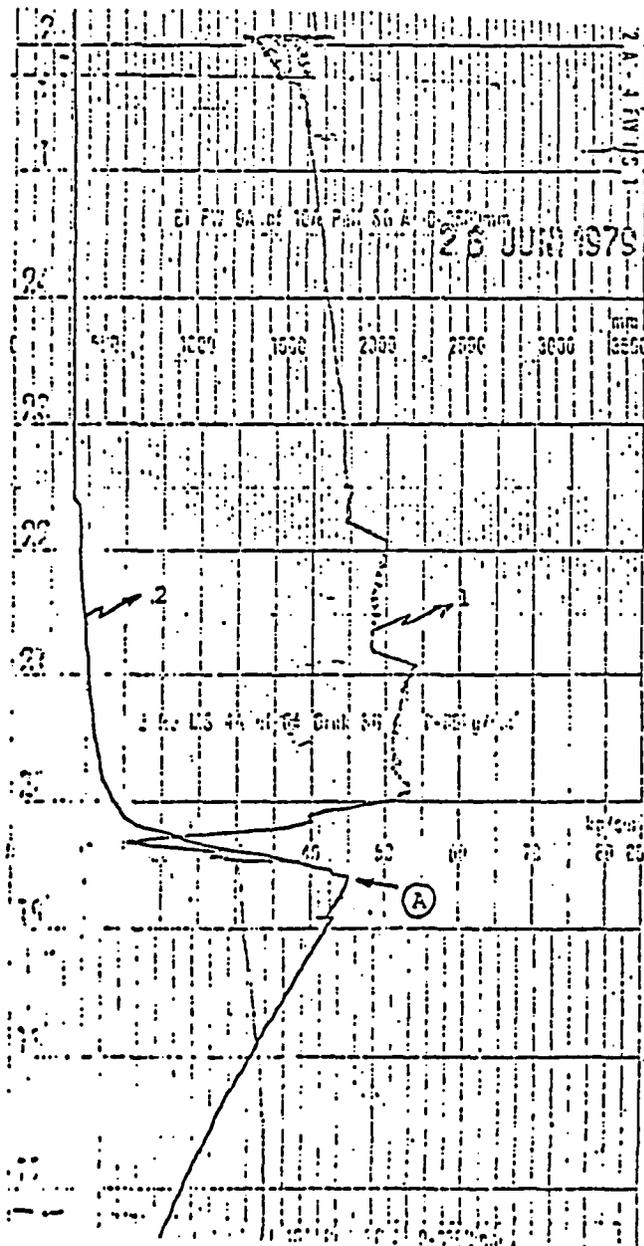
FIGUUR 6

Schrijver : 2 A - 4 FW MS, 1°

- | | | |
|-------------------------|----------------------------------|---------------------------|
| 1. Bl. L. FW 9 A - 10 A | <i>SG A level</i>
Peil SGA | 0 - 3500 mm |
| 2. Ro. P. MS 4 A - 6 A | <i>SG A pressure</i>
Druk SGA | 0 - 85 kg/cm ² |

Meting in dienst spoor 1 :

spoor 2 :



SG B - level and pressure

SG. B PEIL - DRUK

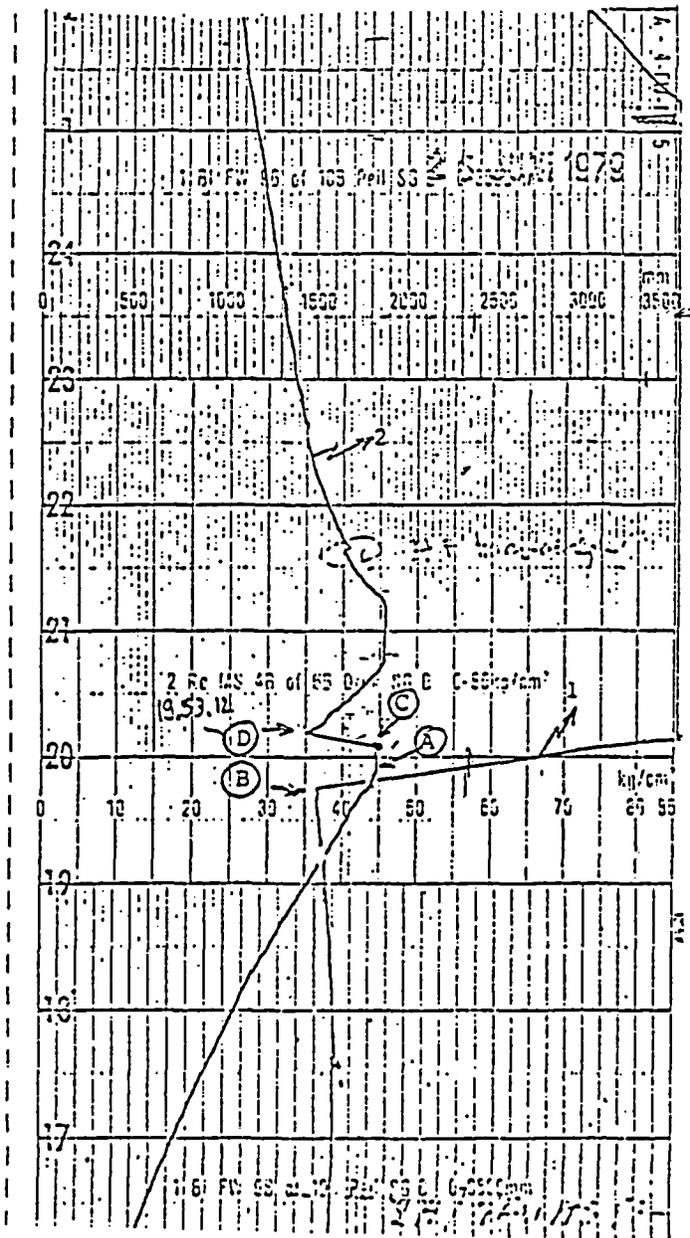
FIGUR 7

Schrijver : 2. A - 4 FW MS 5

- 1. Bl. L. FW 9 B - 10 B *SG B level* Peil SG B 0 - 3500 mm
- 2. Ro. P. MS 4 B - 6 B *SG B pressure* Druk SG B 0 - 85 kg/cm²

Meting in dienst spoor 1 :

. spoor 2 :



* ANNEXE 2 *

=DOEL2 SGTR INCIDENT 25 JUNE 1979

*
*
*
*
*

100 NEW TRANSNT
101 RUN
201 2700. 1.E-6 0.25 3 8 800 2000

* MINOR EDIT *

301	P	330010000	* PRESSURISER
302	CNTRLVAR	031	* CORE POWER
303	CNTRLVAR	305	* HEATERS POWER
304	CNTRLVAR	901	* PRESSU LEVEL
305	MFLOWJ	183000000	* LETDOWN
306	MFLOWJ	181000000	* CV
307	P	123010000	* PRESSURE AT BREAK
308	P	621010000	*
309	VOIDG	045010000	* UPPER HEAD
310	CNTRLVAR	007	*
311	CNTRLVAR	600	*
312	CNTRLVAR	700	*
315	CNTRLVAR	078	* HPSI
318	CNTRLVAR	969	* FW + AFW
319	CNTRLVAR	979	*
320	MFLOWJ	865000000	* STEAM TURBOPUMP
322	P	630010000	* SG PRESSURE
323	P	730010000	*
324	MFLOWJ	617000000	*
325	MFLOWJ	717000000	*
328	CNTRLVAR	610	* HEAT TRANSFER RATE
329	CNTRLVAR	710	*
330	CNTRLVAR	009	* ENERGY BALANCE
331	MFLOWJ	112030000	* SURGELINE MASS FLOWRATE
332	MFLOWJ	618000000	*
333	CNTRLVAR	002	* MASS BALANCE
334	CNTRLVAR	612	* SG B RISER LEVEL
335	CNTRLVAR	712	* SG A
336	MFLOWJ	043010000	* PRIMARY MASS FLOW RATES
337	MFLOWJ	043020000	
338	MFLOWJ	043030000	
339	CNTRLVAR	045	* PRESSURISER MASS

*			
340	MFLOWJ	618000000	* BREAK
341	CNTRLVAR	012	*
342	VOIDGJ	618000000	*
343	TEMPF	015030000	* VESSEL TEMPERATURE
344	TEMPF	041010000	*
345	TEMPF	045010000	*
346	TEMPG	330010000	* PRESSU TEMPERATURE
347	CNTRLVAR	991	* PRESSU HEAT LOSSES
348	CNTRLVAR	113	* AVG TEMPERATURE
349	CNTRLVAR	114	* SUBCOOLING
*			* SGB VOIDG
351	VOIDG	620010000	
352	VOIDG	620020000	
353	VOIDG	620030000	
354	VOIDG	620040000	
355	VOIDG	621010000	
356	VOIDG	623010000	
357	VOIDG	611010000	
358	VOIDG	613010000	
359	VOIDG	612010000	
*			
360	CNTRLVAR	600	* SG B MASS INVENTORY
361	VOIDG	720010000	* SG A VOIDG
362	VOIDG	720020000	
363	VOIDG	720030000	
364	VOIDG	720040000	
365	VOIDG	721010000	
366	VOIDG	723010000	
367	VOIDG	711010000	
368	VOIDG	713010000	
369	VOIDG	712010000	
*			
370	MFLOWJ	623030000	* SG B SEPARATOR MASS FLOW RATES
371	MFLOWJ	623020000	*
372	MFLOWJ	623030000	*
373	MFLOWJ	611010000	* SG B BY PASS MASS FLOW RATES
374	MFLOWJ	611020000	*
375	MFLOWJ	617000000	* SG B DOWNCOMER MASS FLOW RATE
376	CNTRLVAR	611	* SG B COLLAPSED WATER LEVEL
377	CNTRLVAR	613	* SG B MEASURED WATER LEVEL
378	VAPGEN	613010000	
379	VAPGEN	622010000	
*			
380	MFLOWJ	723030000	* SG A SEPARATOR MASS FLOW RATES
381	MFLOWJ	723020000	*
382	MFLOWJ	723030000	*
383	MFLOWJ	711010000	* SG A BYPASS MASS FLOW RATES
384	MFLOWJ	711020000	*
385	MFLOWJ	717000000	* SG A DOWNCOMER MASS FLOW RATE
386	CNTRLVAR	711	* SG A COLLAPSED WATER LEVEL
387	CNTRLVAR	713	* SG A MEASURED WATER LEVEL
388	VAPGEN	713010000	
389	VAPGEN	722010000	
*			

```

391 VOIDG      630010000
392 VOIDG      625010000
393 VOIDG      627010000
394 CNTRLVAR    993
395 TEMPF      251020000
396 TEMPF      211030000
397 TEMPF      151020000
398 TEMPF      113010000
399 CPUTIME     0

```

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*
*
*

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*****
* TRIP
*****

```

```

*
*
*
550 TIME      O GT NULL O      900      N      * OPEN RELIEF VALVE
*
561 TIME      O LT NULL O      1900.    N      * TURBOPUMP ON
*
*
503 TIME      O      GT      NULL    O      1112.    N      * PUMP B STOP
504 TIME      O      LT      NULL    O      2105.    N      * PUMP B STOP
529 TIME      O      GT      NULL    O      2106.    N      * SPRAY
530 TIME      O      LT      NULL    O      2287.    N      * SPRAY
542 TIME      O      GT      NULL    O      2248.    N      * HEATER ON
560 TIME      O      GT      NULL    O      1482.    N      * TURBOPUMP
571 P 330010000 LT      NULL    O      117.E5   L      * SI
580 CNTRLVAR 901 LT      NULL    O      0.2      L      * LPR 20%
574 TIME      O      GT      NULL    O      1428.    N      * MOTOPUMP "A" ON
575 TIME      O      GT      NULL    O      1479.    N      * MOTOPUMP "B"
576 TIME      O      LT      NULL    O      1992.    N      * "B"
591 TIME      O      GT      NULL    O      1000.    N      * SGA DISCHARGE
*

```

```

601 503 AND 504 N
602 -503 OR -504 N
630 529 AND 530 N * SPRAY ON
641 -591 AND -580 N *
642 641 OR 542 N * HEATER ON
661 560 AND 561 N * TURBOPUMP ON
676 575 AND 576 N * MOTOPUMP B ON
*

```

```

598 TIME      O LE NULL O      10000.   L      * BIDON
599 TIME      O GE NULL O      10000.   L      * BIDON
*
*
*

```

```

*****
* VESSEL
*****
*
*

```

173


```

0310000 CORE PIPE
0310001 4
0310101 2.5 4
0310301 0.6095 4
0310601 90. 4
0310801 4.E-5 0.0155 4
0310901 2.5 2.5 3
0311001 00 4
0311101 0000 3
0311300 1
*
* CORE EXIT
*
* CORE BYPASS
*
0350000 CORBYP5 SNGLVOL
0350101 0. 2.438 1.9706 0. 90. 2.438 4.E-5 0.0 00
*
0390000 COREX BRANCH
0390001 3 1
0390101 0. 0.6085 2.3252 0. 90. 0.6085 4.E-5 0. 00
0391101 031010000 039000000 2.5 1.8 1.8 0000
0392101 035010000 039000000 0.1 6.32 6.32 0000
0393101 039010000 041000000 1. 0.262 0.262 0000
*
* CORE PLENUM
*
0410000 CORPLEN SNGLVOL
0410101 0. 0.8965 5.103 0. 90. 0.8965 4.E-5 0.2 00
*
* UPPER PLENUM
*
0430000 UPPLEN BRANCH
0430001 3 1
0430101 0 1.584 9.349 0. 90. 1.584 4.E-5 0.2 00
0431101 041010000 043000000 0. 0. 0. 0000
0432101 043000000 111000000 0.383 0.1737 0.1737 0000
0433101 043000000 211000000 0.383 0.1737 0.1737 0000
*
* VESSL DOME
*
0450000 DOME BRANCH
0450001 1 1
0450101 0. 1.496 8.833 0. 90. 1.496 4.E-5 0. 00
0451101 043010000 045000000 0.00256 2.7 2.7 0000
*
* HEAT STRUCTURES
*
* CORE HEAT STRUCTURES
*
10311000 4 5 2 1 0.
10311100 0 1
10311101 2 0.004647 1 0.0047425 1 0.005395
10311201 1 1 1 2 -4 3 -2 4
10311301 1. 1 1. 2 0. 3 0. 4
    
```

10311400 -1
 10311401 1158.3 1061.4 812.12 606. 583.
 10311402 1158.3 1061.4 812.12 606. 583.
 10311403 1158.3 1061.4 812.12 606. 583.
 10311404 1158.3 1061.4 812.12 606. 583.
 10311501 0 0 0 0 0. 4
 10311601 031010000 10000 1 0 441.0 4
 10311701 100 0.16667 0. 0. 1
 10311702 100 0.33333 0. 0. 2
 10311703 100 0.33333 0. 0. 3
 10311704 100 0.16667 0. 0. 4
 10311901 0 0. 0.0136 0.0136 4

*
 *

 * FIRST PRIMARY LOOP

* HOT LEG

*
 1110000 HTLG11 SNGLVOL
 1110101 0.383 2.476 0. 0. 0. 4.E-5 0.698 00
 *
 1120000 HTLG12 BRANCH
 1120001 3 1
 1120101 0.383 2.476 0. 0. 0. 4.E-5 0.698 00
 1121101 111010000 112000000 0. 0. 0. 0000
 1122101 112010000 113000000 0. 0. 0. 0000
 1123101 301000000 112000000 0. 10. 10. 0000 *1 KF TO BE CHECKED

*
 *
 1130000 HTLG13 SNGLVOL
 1130101 0.383 2.476 0. 0. 30.8 0.95 4.E-5 0.698 00

* STEAM GENERATOR B

* INLET BOX

*
 1210000 INBOX BRANCH
 1210001 2 1
 1210101 0. 1.685 4.392 0. 90. 1.685 4.E-5 1.822 00
 1212101 121010000 122000000 0.99 0.5 1.0 0000
 1211101 113010000 121000000 0.435 0.4145 0.5 0000
 *
 1220000 UUTUBE PIPE
 1220001 8
 1220101 0.99 8
 1220301 0.908 8
 1220601 90. 8
 1220801 1.6E-6 0.01966 8
 1221001 00 8
 1221101 0000 7

```

1221300 1
*
*
1230000 UJUN BRANCH
1230001 2 1
1230101 0.99 1.816 0. 0. 45.0 1.284 1.6E-6 0.01966 00
1231101 122010000 123000000 00.00 0. 0. 0000
1232101 123010000 124000000 00.00 0. 0. 0000
*
*
*
1240000 DUTUB1 PIPE
1240001 5
1240101 0.99 5
1240301 1.816 5
1240601 -45. 1
1240602 -90. 5
1240801 1.E-7 0.01966 5
1241001 00 5
1241101 0000 4
1241300 1
*
* OUBOX BRANCH
*
1250000 OUBOX BRANCH
1250001 2 1
1250101 0. 1.685 4.392 0. -90. -1.685 4.E-5 1.822 00
1251101 124010000 125000000 0.99 1. 0.5 0000
1252101 125010000 131000000 0.487 0.040 1.000 0000
*
*
* STEAM GENERATOR SECONDARY SIDE
*
*
*
6110000 BYPASSB BRANCH
6110001 2 0
6110101 8.048 2.424 0. 0. 90. 2.424 5.E-5 0. 00
6112101 611010000 627000000 0.00 00.0 00.0 00000
6111101 611000000 612000000 0. 200. 1000. 00000
*
6120000 B612 BRANCH
6120001 1 0
6120101 8.048 .735 0. 0. -90. -.735 5.E-5 0. 00
6121101 612010000 613000000 00.00 0. 200. 0000
*
6130000 FEEDMIX SNGLVOL
6130101 0. 0.697 5.596 0. -90. -0.697 5.E-5 0. 00
*
6140000 FEEDDWC SNGLJUN
6140101 613010000 615000000 8.0398 0. 1000. 0000
*
6150000 SGLDCOME1 ANNULUS
6150001 4

```

```

6150101 2.589 1
6150102 0.4845 4
6150301 1.938 1
6150302 2.55 4
6150601 -90. 4
6150801 5.E-5 0. 1
6150802 5.E-5 0.1015 4
6150901 0. 1000. 3
6151001 00 4
6151101 0000 3
6151300 0
*
6170000 SGDCPJ1 SNGLJUN
6170101 615010000 620000000 .000 3.3 1000. 0000
*
*
*
6200000 SQPIPE PIPE
6200001 4
6200101 4.432 4
6200301 1.816 4
6200801 90. 4
6200801 1.E-4 0.03543 4
6201001 00 4
6201101 0000 3
6201300 0
*
*
6210000 BREAK BRANCH
6210001 2 0
6210101 4.432 2.324 0. 0. 90.0 2.324 1.E-4 0.03543 00
6211101 620010000 621000000 00.00 0. 0. 0000
6212101 621010000 622000000 00.00 0. 0. 0000
*
*
*
6220000 TOPRI1 BRANCH
6220001 0 0
6220101 4. 1.432 0. 0. 90.0 1.432 0.00005 0. 00
*
*
6230000 TOPRI2 SEPARATR
6230001 3 0
6230101 4. 2.424 0. 0. 90.0 2.424 0.00005 0. 00
6231101 623010000 625000000 2.75 20.4 20.4 01000 0.1
6232101 623000000 612000000 0.00 1.0 1000. 01000 0.1
6233101 622010000 623000000 0.00 1.81 1.81 01000
*
*-----*
* B R E A K J U N C T I O N *
*-----*
6180000 BREAK SNGLJUN
6180101 122010000 621000000 1.724E-4 2.6 2.6 00003 0. 0.
6180201 1 14.987 0. 0.
*

```

```

*
*
6300000 SGDOME1 BRANCH
6300001 1 1
6300101 12.5600 2.093 0. 0. 90. 2.093 5.E-5 0. 00
6301101 630010000 640000000 .13 .2269 .4969 0000

```

```

*
*
6250000 DRYER BRANCH
6250001 1 1
6250101 4.00 1. 0. 0. 90. 1. 5.E-5 0. 00
6251101 625010000 630000000 .00 .0000 .0000 00000

```

```

*
*
6270000 PLENUMB BRANCH
6270001 1 1
6270101 8.5600 1. 0. 0. 90. 1. 5.E-5 0. 00
6271101 627010000 625000000 .00 .0000 .0000 00003

```

```

*
*
6400000 SGL1 PIPE * STEAM LINES
6400001 3
6400101 0.288 3
6400301 22.7171 3
6400601 00.0 3
6400801 4.E-5 00.000 3
6401001 00 3
6401101 0000 2
6401300 1

```

```

*
*
*
*
8600000 TV860 TMDPVOL
8600101 10. 1. 0. 0 0. 0. 0. 0. 00
8600200 2
8600201 0. 1.E5 1.

```

```

*
*
8650000 TJ865 TMDPJUN * TURBOPUMP (STEAM)
8650101 640010000 860000000 0.
8650200 1 661 CNTRLVAR 883
8650201 -1. 0. 0. 0.0
8650202 0. 0. 0. 0.0
8650203 10. 0. 10. 0.0

```

* AUXILIARY FEED WATER (1MTPUMP)

```

*
*
6050000 POOL1 TMDPVOL
6050101 1. 1. 0. 0. 0. 0. 0. 0. 00
6050200 3
6050201 0. 1.E5 308.15

```

```

*
*
6060000 POOL2 TMDPVOL
6060101 1. 1. 0. 0. 0. 0. 0. 0. 00
6060200 3

```

```

6060201 0. 1.E5 308.15
*
6070000 MTPOM TMDPJUN
6070101 60500000 613000000 0.068
6070200 1 676 P 630010000
6070201 -1. 0. 0. 0.
6070202 1.E5 27.71 0. 0.
6070203 5.E5 27.17 0. 0.
6070204 10.E5 26.46 0. 0.
6070205 15.E5 25.74 0. 0.
6070206 20.E5 24.99 0. 0.
6070207 25.E5 24.22 0. 0.
6070208 30.E5 23.42 0. 0.
6070209 35.E5 22.58 0. 0.
6070210 40.E5 21.72 0. 0.
6070211 45.E5 20.81 0. 0.
6070212 50.E5 19.86 0. 0.
6070213 55.E5 18.85 0. 0.
6070214 60.E5 17.78 0. 0.
6070215 65.E5 16.63 0. 0.
6070216 70.E5 15.39 0. 0.
6070217 75.E5 14.02 0. 0.
6070218 80.E5 12.46 0. 0.
6070219 85.E5 10.64 0. 0.

```

```

*IDELIVERY CURVE OF 1
* MOTOPUMP FEEDING 150
*

```

* STEAM GENERATOR HEAT STRUCTURE

```

*
*
16202000 15 2 2 1 0.00983
16202100 0 1
16202101 1 0.0111
16202201 6 1
16202301 0. 1
16202501 122010000 10000 1 0 164.04455 8
16202502 123010000 0 1 0 164.04455 10
16202503 124010000 10000 1 0 328.08909 15
16202601 620010000 0000 1 0 185.2385 2
16202602 620020000 0000 1 0 185.2385 4
16202603 620030000 0000 1 0 185.2385 6
16202604 620040000 0000 1 0 185.2385 8
16202605 621010000 0000 1 0 185.2385 10
16202607 621010000 0000 1 0 370.477 11
16202608 620040000 -10000 1 0 370.477 15
16202701 0 0. 0. 0. 0. 15
16202801 0. 0.01966 0. .908 10
16202802 0. 0.01966 0. 1.816 15
16202901 0. 0.03543 0.02223 .908 10
16202902 0. 0.03543 0.02223 1.816 15

```

* PUMP SUCTION LEG

*
*
*
1310000 B131 BRANCH

```

1310001 1 1
1310101 0.487 1.806 0. 0. -90. -1.806 4.E-5 .787 00
1311101 131010000 1320000000 .487 0. 0. 0000
*
1320000 B132 BRANCH
1320001 1 1
1320101 0.487 1.806 0. 0. -90. -1.806 4.E-5 .787 00
1321101 132010000 1330000000 .487 0.11 0.11 0000
*
1330000 PSUC1 PIPE
1330001 2
1330101 0.487 2
1330301 1.633 1
1330302 1.43 2
1330601 0. 1
1330602 90. 2
1330801 4.E-5 0.787 2
1330902 0.11 0.11 1
1331001 00 2
1331101 0000 1
1331300 1
*
*
*
* PUMP
*
1400000 PUMP1 PUMP
1400101 0. 1.232 7.757 0. 90. 1.232 00
1400108 133010000 0.487 0.4385 0.4385 0000
1400109 151000000 0.383 0.2712 0.2712 0000
1400301 -2 -1 -2 -1 0 0 1
1400302 103.67 1. 4.826 58. 19998. 3458.75 755. 23536.
1400303 -862.87 4064.6 114.6 204.3
1406100 0
1406101 0. 103.67
1406102 1112. 103.67
1406103 1117. 77.75
1406104 1122. 69.46
1406105 1127. 45.1
1406106 1132. 35.25
1406107 1142. 20.73
1406108 1152. 8.81
1406109 1172. 0.
1406110 2106. 0.
1406111 2124. 69.46
1406112 2133. 103.67
1406113 9999. 103.67
*
*
*
* COLD LEG
*
1510000 CLDLG11 PIPE
1510001 2
1510101 0.383 2

```

```

1510301 1.968 2
1510601 0. 2
1510801 4.E-5 0.698 2
1511001 00 2
1511101 0000 1
1511300 1
*
1520000 CLDLG12 BRANCH
1520001 2 1
1520101 0.383 1.968 0. 0. 0. 0. 4.E-5 0.698 00
1521101 151010000 152000000 0.383 0. 0. 0000
1522101 152010000 153000000 0.383 0. 0. 0000
*
1530000 CLDLG13 SNGLVOL
1530101 0.383 1.968 0. 0. 0. 0. 4.E-5 0.698 00
*
*
* ECCS CLDLG B
*
1700000 SIPPOOL TMDPVOL
1700101 100. 10. 0. 0. 0. 0. 0. 0. 11
1700200 3
1700201 0. 1.E5 288.15
*
*
*
* ECCS-4PP (50%)
*
*
1710000 ECCSJUN TMDPJUN
1710101 170000000 152000000 0.1
1710200 1 571 CNTRLVAR 070 * DELIVERY CURVE OF 4 PUMPS
1710222 107.91E5 0.00 0. 0.
1710221 105.00E5 10.85 0. 0.
1710220 100.00E5 17.00 0. 0.
1710219 95.00E5 21.40 0. 0.
1710218 90.00E5 24.80 0. 0.
1710217 85.00E5 28.00 0. 0.
1710216 80.00E5 30.90 0. 0.
1710215 75.00E5 33.60 0. 0.
1710214 70.00E5 36.25 0. 0.
1710213 65.00E5 38.90 0. 0.
1710212 60.00E5 41.30 0. 0.
1710211 55.00E5 43.65 0. 0.
1710210 50.00E5 46.00 0. 0.
1710209 45.00E5 48.25 0. 0.
1710208 40.00E5 50.40 0. 0.
1710207 35.00E5 52.40 0. 0.
1710206 30.00E5 54.40 0. 0.
1710205 25.00E5 56.30 0. 0.
1710204 20.00E5 58.00 0. 0.
1710203 15.00E5 60.00 0. 0.
1710202 13.12E5 61.00 0. 0.
1710201 -1. 0. 0. 0.

```

*
*
*
*
*

1800000 CVV TMDPVOL ***
 1800101 10. 1. 0. 0 0. 0: 0. 0. 00
 1800200 3 580
 1800201 -1. 155.E5 523.
 1800202 0. 155.E5 293.15

*
*

1810000 CVJ TMDPJUN * CHARGING FLOW
 1810101 180000000 152000000 1.E-1
 1810200 1
 1810201 0.0 3.2 0.0 0.0
 1810202 170.0 4.08 0.0 0.0
 1810203 171. 8.16 0. 0.
 1810204 900. 8.16 0. 0.
 1810205 901. 12.25 0. 0.
 1810206 9999. 12.25 0. 0.

*
*
*

1820000 LDV TMDPVOL
 1820101 10. 1. 0. 0 0. 0. 0. 0. 00
 1820200 3
 1820201 0. 1.E5 293.

*
*
*

1830000 LDJ TMDPJUN * LETDOWN
 1830101 132010000 182000000 1.E-1
 1830200 1 580
 1830201 -1. 6.8 0. 0.
 1830202 0. 0. 0. 0.

*
*
*

 * SECOND PRIMARY LOOP

* HOT LEG

2110000 HTLG20 PIPE
 2110001 3
 2110101 0.383 3
 2110301 2.476 3
 2110601 0. 2
 2110602 30.8 3
 2110701 0. 2
 2110702 0.95 3

```

2110801 4.E-5 0.698 3
2111001 00 3
2111101 0000 2
2111300 1
*
* INLET BOX
*
2210000 INBOX BRANCH
2210001 2 1
2210101 0. 1.685 4.392 0. 90. 1.685 4.E-5 1.822 00
2212101 221010000 222000000 0.99 0.5 1.0 0000
2211101 211010000 221000000 0.435 0.4145 0.5 0000
*
2220000 UTUB1 PIPE
2220001 10
2220101 0.99 10
2220301 0.908 10
2220601 90. 8
2220602 45. 10
2220801 1.E-7 0.01966 10
2221001 00 10
2221101 0000 9
2221300 1
*
2230000 UJUN1 SNGLJUN
2230101 222010000 224000000 0.99 0. 0. 0000 0. 0.
*
2240000 DUTUB1 PIPE
2240001 5
2240101 0.99 5
2240301 1.816 5
2240601 -45. 1
2240602 -90. 5
2240801 1.E-7 0.01966 5
2241001 00 5
2241101 0000 4
2241300 1
*
* OUBOX BRANCH
*
2250000 OUBOX BRANCH
2250001 2 1
2250101 0. 1.685 4.392 0. -90. -1.685 4.E-5 1.822 00
2251101 224010000 225000000 0.99 1. 0.5 0000
2252101 225010000 231000000 0.487 0.040 1.000 0000
*
* STEAM GENERATOR SECONDARY SIDE
*
7110000 BYPASSB BRANCH
7110001 2 0
7110101 8.048 2.424 0. 0. 90. 2.424 5.E-5 0. 00
7112101 727000000 711010000 0.00 0.00 10.0 100.0 00000
7111101 711000000 712000000 0. 10. 1000. 00000

```

```

*
7120000 B712 BRANCH
7120001 1 0
7120101 8.048 .735 0. 0. -90. -.735 5.E-5 0. 00
7121101 712010000 713000000 00.00 0. 200. 0000
*
7130000 FEEDMIX SNGLVCL
7130101 0. 0.697 5.596 0. -90. -0.697 5.E-5 0. 00
*
7140000 FEEDDWC SNGLJUN
7140101 713010000 715000000 8.0398 0. 1000. 0000
*
7150000 SGLDCOME1 ANNULUS
7150001 4
7150101 2.589 1
7150102 0.4845 4
7150301 1.938 1
7150302 2.55 4
7150601 -90. 4
7150801 5.E-5 0. 1
7150802 5.E-5 0.1015 4
7150901 0. 1000. 3
7151001 00 4
7151101 0000 3
7151300 0
*
7170000 SGDCPJ1 SNGLJUN
7170101 715010000 720000000 .000 3.3 1000. 0000
*
*
*
7200000 SGPIPE PIPE
7200001 4
7200101 4.432 4
7200301 1.816 4
7200601 90. 4
7200801 1.E-4 0.03543 4
7201001 00 4
7201101 0000 3
7201300 0
*
*
7210000 BREAK BRANCH
7210001 2 0
7210101 4.432 2.324 0. 0. 90.0 2.324 1.E-4 0.03543 00
7211101 720010000 721000000 00.00 0. 0. 0000
7212101 721010000 722000000 00.00 0. 0. 0000
*
*
*
7220000 TOPRI1 BRANCH
7220001 0 1
7220101 4. 1.432 0. 0. 90.0 1.432 0.00005 0. 00
*

```

```

7230000 TOPRI2 SEPARATR
7230001 3 0
7230101 4. 2.424 0. 0. 90.0 2.424 0.00005 0. 00
7231101 723010000 725000000 2.75 20.4 20.4 01000 0.1
7232101 723000000 712000000 0.00 1.0 1000. 01000 0.1
7233101 722010000 723000000 0.00 1.81 1.81 01000
*
*
*
7300000 SGDOME1 BRANCH
7300001 1 1
7300101 12.5600 2.093 0. 0. 90. 2.093 5.E-5 0. 00
7301101 730010000 740000000 .13 .2269 .4969 0000
*
7250000 DRYER BRANCH
7250001 1 1
7250101 4.00 1. 0. 0. 90. 1. 5.E-5 0. 00
7251101 725010000 730000000 .00 .0000 .0000 00000
*
7270000 PLENUMB BRANCH
7270001 1 1
7270101 8.5600 1. 0. 0. 90. 1. 5.E-5 0. 00
7271101 727010000 725000000 .00 .0000 .0000 00003
*
*
7400000 SGL1 PIPE
7400001 3
7400101 0.288 3
7400301 22.7171 3
7400601 00.0 3
7400801 4.E-5 00.000 3
7401001 00 3
7401101 0000 2
7401300 1
*
*
*
7940000 ATM TMDPVOL
7940101 10. 10. 0. 0. 0. 0. 0. 00
7940200 2
7940201 0.0 1.0E5 1.
*
*
7910000 RELIEF VALVE
7910101 740010000 794000000 2.74670E-3 1. 1. 0100 0. 0.
7910201 1 0. 0. 0.
7910300 MTRVLV
7910301 550 599 3.33E-3 0.
*
*
*
AUXILIARY FEED WATER (1MTPUMP)
*
7050000 POOLMT TMDPVOL
7050101 1. 1. 0. 0. 0. 0. 0. 00
    
```

```

7050200 3
7050201 0. 1.E5 308.15
*
7060000 TURPOOL TMDPVOL
7060101 1. 1. 0. 0. 0. 0. 0. 00
7060200 3
7060201 0. 1.E5 308.15 *! DELIVERY CURVE OF 1
* * MOTOPUMP FEEDING 1SG
7070000 AFWMT TMDPJUN
7070101 705000000 713000000 0.136
7070200 1 574 P 730010000
7070201 -1. 0. 0. 0.
7070202 1.E5 27.71 0. 0.
7070203 5.E5 27.17 0. 0.
7070204 10.E5 26.46 0. 0.
7070205 15.E5 25.74 0. 0.
7070206 20.E5 24.99 0. 0.
7070207 25.E5 24.22 0. 0.
7070208 30.E5 23.42 0. 0.
7070209 35.E5 22.58 0. 0.
7070210 40.E5 21.72 0. 0.
7070211 45.E5 20.81 0. 0.
7070212 50.E5 19.86 0. 0.
7070213 55.E5 18.85 0. 0.
7070214 60.E5 17.78 0. 0.
7070215 65.E5 16.63 0. 0.
7070216 70.E5 15.39 0. 0.
7070217 75.E5 14.02 0. 0.
7070218 80.E5 12.46 0. 0.
7070219 85.E5 10.64 0. 0.
*
*
* STEAM GENERATOR HEAT STRUCTURE
*
17202000 15 2 2 1 0.00983
17202100 0 1
17202101 1 0.0111
17202201 6 1
17202301 0. 1
17202501 222010000 10000 1 0 164.04455 10
17202502 224010000 10000 1 0 328.08909 15
17202601 720010000 0000 1 0 185.2385 2
17202602 720020000 0000 1 0 185.2385 4
17202603 720030000 0000 1 0 185.2385 6
17202604 720040000 0000 1 0 185.2385 8
17202605 721010000 0000 1 0 185.2385 10
17202607 721010000 0000 1 0 370.477 11
17202608 720040000 -10000 1 0 370.477 15
17202701 0 0. 0. 0. 15
17202801 0. 0.01966 0. .908 10
17202802 0. 0.01966 0. 1.816 15
17202901 0. 0.03543 0.02223 .908 10
17202902 0. 0.03543 0.02223 1.816 15
*

```

*
* PUMP SUCTION LEG
*

2310000 PSUC2 PIPE
2310001 4
2310101 0.487 4
2310301 1.806 2
2310302 1.633 3
2310303 1.430 4
2310602 -90. 2
2310603 0. 3
2310604 90. 4
2310801 4.E-5 0.787 4
2310901 0. 0. 1
2310902 0.11 0.11 3
2311001 00 4
2311101 0000 3
2311300 1

* PUMP
*

2400000 PUMP2 PUMP
2400101 0. 1.232 7.757 0. 90. 1.232 00
2400108 231010000 0.487 0.4385 0.4385 0000
2400109 251000000 0.383 0.27120 0.27120 0000
2400301 -2 -1 -2 -1 -1 0 1
2400302 103.67 1. 4.828 58. 19998. 3458.75 755. 23538.
2400303 -862.87 4064.6 114.6 204.3

*CHECK

* COLD LEG
*

2510000 CLDLG21 PIPE
2510001 2
2510101 0.383 2
2510301 1.968 2
2510601 0. 2
2510801 4.E-5 0.698 2
2511001 00 2
2511101 0000 1
2511300 1

* COLD LEG
*

2520000 CLDLG22 BRANCH
2520001 2 1
2520101 0.383 1.968 0. 0. 0. 0. 4.E-5 0.698 00
2521101 251010000 252000000 0.383 0. 0. 0000
2522101 252010000 253000000 0.383 0. 0. 0000

* COLD LEG
*

2530000 CLDLG23 SINGLVOL
2530101 0.383 1.968 0. 0. 0. 0. 4.E-5 0.698 00
*
* ECCS COLD LEG A
*

```

2700000 SIPPOOL TMDPVOL
2700101 100. 10. 0. 0. 0. 0. 0. 11
2700200 3
2700201 0. 1.E5 288.15

```

```

*
*
* ECCS-4PP (50%)
*

```

```

2710000 ECCSJUN TMDPJUN
2710101 270000000 252000000 0.1
2710200 1 571 CNTRLVAR 070 * DELIVERY CURVE OF 4 PUMPS
2710222 107.91E5 0.00 0. 0.
2710221 105.00E5 10.65 0. 0.
2710220 100.00E5 17.00 0. 0.
2710219 95.00E5 21.40 0. 0.
2710218 90.00E5 24.80 0. 0.
2710217 85.00E5 28.00 0. 0.
2710216 80.00E5 30.90 0. 0.
2710215 75.00E5 33.60 0. 0.
2710214 70.00E5 36.25 0. 0.
2710213 65.00E5 38.90 0. 0.
2710212 60.00E5 41.30 0. 0.
2710211 55.00E5 43.65 0. 0.
2710210 50.00E5 46.00 0. 0.
2710209 45.00E5 48.25 0. 0.
2710208 40.00E5 50.40 0. 0.
2710207 35.00E5 52.40 0. 0.
2710206 30.00E5 54.40 0. 0.
2710205 25.00E5 56.30 0. 0.
2710204 20.00E5 58.00 0. 0.
2710203 15.00E5 60.00 0. 0.
2710202 13.12E5 61.00 0. 0.
2710201 -1. 0. 0. 0.

```

```

*
*
*****
* PRESSURIZER
*****

```

```

3010000 SURGELN PIPE
3010001 3
3010101 0.03875 3
3010301 6.3465 2
3010302 4.07 3
3010601 0. 2
3010602 90. 3
3010801 4.E-5 0. 3
3010901 0.294 0.294 2
3011001 00 3
3011101 0000 2
3011300 1
*
3050000 PRESLL SNGLJUN

```

```

3050101 301010000 310000000 0. 2. 2. 0100 0. 0.
*
3100000 PRES310 BRANCH
3100001 0 1
3100101 1.6964 0.3 0. 0. 90. 0.3 0. 0. 01
3100200 2 155.E5 0.
*
3110000 PRES311 BRANCH
3110001 1 1
3110101 1.6964 0.6 0. 0. 90. 0.6 0. 0. 01
3110200 2 155.E5 0.
3111101 310010000 311000000 0. 0. 0. 0000
3111201 0. 0. 0.
*
3120000 PRES312 BRANCH
3120001 1 1
3120101 2.5164 1.766 0. 0. 90. 1.766 0. 0. 01
3120200 2 155.E5 0.
3121101 311010000 312000000 0. 0. 0. 0000
3121201 0. 0. 0.
*
3130000 PRES313 BRANCH
3130001 1 1
3130101 2.5164 1.766 0. 0. 90. 1.766 0. 0 01
3130200 2 155.E5 0.25751
3131101 312010000 313000000 0. 0. 0. 0000
3131201 0. 0. 0.
*
3140000 PRES314 BRANCH
3140001 1 1
3140101 2.5164 1.539 0. 0. 90. 1.539 0. 0. 01
3140200 2 155.E5 1.
3141101 313010000 314000000 0. 0. 0. 0000
3141201 0. 0. 0.
*
3150000 PRES315 BRANCH
3150001 1 1
3150101 2.5164 1.515 0. 0. 90. 1.515 0. 0. 01
3150200 2 155.E5 1.
3151101 314010000 315000000 0. 0. 0. 0000
3151201 0. 0. 0.
*
3160000 PRES316 BRANCH
3160001 1 1
3160101 2.5164 0.8 0. 0. 90. 0.8 0. 0. 01
3160200 2 155.E5 1.
3161101 315010000 316000000 0. 0. 0. 0000
3161201 0. 0. 0.
*
3170000 PRES317 BRANCH
3170001 2 1
3170101 2.5164 0.6 0. 0. 90. 0.6 0. 0. 01
3170200 2 155.E5 1.
3171101 316010000 317000000 0. 0. 0. 0000
3172101 317010000 320000000 0. 0. 0. 0000

```

```

3171201 0. 0. 0.
3172201 0. 0. 0.
*
3200000 PRES320 PIPE
3200001 6
3200101 2.5164 1 1.6964 6
3200301 0.4 1 0.3 2 0.2 3 0.15 4 0.1 6
3200601 90. 6
3200801 0. 0. 6
3201001 01 6
3201101 0000 5
3201201 2 155.E5 1. 0. 0. 0. 6
3201300 1
3201301 0. 0. 0. 5
*
3300000 PRESDOM BRANCH
3300001 1 1
3300101 1.6964 0.05 0. 0. 90. 0.05 0. 0. 01
3300200 2 155.E5 1.
3301101 320010000 330000000 0. 0. 0. 0000
3301201 0. 0. 0.

```

```

*
*
*
*

```

```

* HEATERS

```

```

13113000 2 3 2 1 0.
13113100 0 2
13113101 0.005 1 0.00338 2
13113201 5 2
13113301 1. 1 0. 2
13113401 618. 3
13113501 0 0 0 1 62.532 1
13113502 0 0 0 1 93.798 2
13113601 312010000 0 1 1 62.532 1
13113602 312010000 0 1 1 93.798 2
13113701 311 0.4 0. 0. 1
13113702 311 0.6 0. 0. 2
13113901 0 0. 0. 0.772 1
13113902 0 0. 0. 1.158 2

```

```

*****
* SPRAY SYSTEM
*****

```

```

*
*
*
3410000 SPRJUN SNGLJUN
3410101 152010000 343000000 8.171E-3 0. 0. 0000
3410201 0 0. 0. 0.
*
3430000 SPRLINE PIPE
3430001 4
3430101 8.171E-3 4

```

```

3430301 3.      3
3430302 3.956   4
3430601 90.     4
3430801 4.E-5   0.   4
3431001 00 4
3431101 0000   3
3431201 3 154.88E5 528. 0. 0. 0. 1
3431202 3 154.64E5 528. 0. 0. 0. 2
3431203 3 154.41E5 528. 0. 0. 0. 3
3431204 3 154.13E5 528. 0. 0. 0. 4
3431300 1
3431301 0. 0. 0. 3
*
3450000 RESPR  TMDPJUN
3450101 343010000 317010000 0.00456
3450200 0 630
3450201 -1. 0.0   0. 0.
3450202 0. 2.741 0. 0.
*
3420000 SPRJUN  SNGLJUN
3420101 252010000 344000000 8.171E-3 0. 0. 0000
3420201 0 0. 0. 0.
*
3440000 SPRLINE  PIPE
3440001 4
3440101 8.171E-3 4
3440301 3.      3
3440302 3.956   4
3440601 90.     4
3440801 4.E-5   0.   4
3441001 00 4
3441101 0000   3
3441201 3 154.88E5 528. 0. 0. 0. 1
3441202 3 154.64E5 528. 0. 0. 0. 2
3441203 3 154.41E5 528. 0. 0. 0. 3
3441204 3 154.13E5 528. 0. 0. 0. 4
3441300 1
3441301 0. 0. 0. 3
*
3460000 RESPR  TMDPJUN
3460101 344010000 317010000 0.00456
3460200 0 630
3460201 -1. 0.0   0. 0.
3460202 0. 2.741 0. 0.
*
*****
* CONTAINEMENT
*****
*
9500000 V950  SNGLVOL
9500101 10000. 10.33 0. 0. -90.0 -10.33 4.E-5 0. 00
9500200 1 318.  .01

```

*
 9600000 V960 SNGLVOL
 9600101 10000. 16.53 0. 0. -90.0 -16.53 4.E-5 0. 00
 9600200 1 318. .01
 *

9700000 V970 SNGLVOL
 9700101 10000. 16.53 0. 0. -90.0 -16.53 4.E-5 0. 00
 9700200 1 318. .01
 *

 * HEAT STRUCTURES *

* PRESSURIZER

* SURGE LINE

13000000 3 3 2 1 0.10795
 13000100 0 2
 13000101 0.01 1 0.01858 2
 13000201 3 2
 13000301 0. 2
 13000501 301010000 10000 1 1 6.3465 2
 13000502 301030000 0 1 1 4.07 3
 13000601 0 0 0 1 6.3465 2
 13000602 0 0 0 0 1 4.07 3
 13000701 0 0. 0. 0. 3
 13000801 0 0. 0. 0. 2
 13000802 0 0. 0. 4.07 3

* BOTTOM

13001000 2 4 3 1 0.9
 13001100 0 2
 13001101 0.01 1 0.02 2 0.044 3
 13001201 5 3
 13001301 0. 3
 13001401 617.91 4
 13001501 310010000 0 1 1 0.16666 1
 13001502 311010000 0 1 1 0.33333 2
 13001601 950010000 0 3950 1 0.16666 1
 13001602 950010000 0 3950 1 0.33333 2
 13001701 0 0. 0. 0. 2
 13001801 0 0. 0. 0.3 1
 13001802 0 0. 0. 0.6 2

* CYLINDRICAL SHELL

13002000 7 4 2 1 0.895
 13002100 0 2
 13002101 0.01 1 0.02 2 0.069 3
 13002201 5 3
 13002301 0. 3
 13002401 617.91 4

13002501	312010000	1000000		1	1	1.766	2
13002502	314010000	0		1	1	1.539	3
13002503	315010000	0		1	1	1.515	4
13002504	316010000	0		1	1	0.8	5
13002505	317010000	0		1	1	0.6	6
13002506	320010000	0		1	1	0.4	7
13002601	950010000	0	3950		1	1.766	2
13002602	950010000	0	3950		1	1.539	3
13002603	950010000	0	3950		1	1.515	4
13002604	950010000	0	3950		1	0.8	5
13002605	950010000	0	3950		1	0.6	6
13002606	950010000	0	3950		1	0.4	7
13002701	0 0. 0.	0. 7					
13002801	0 0. 0.	1.766	2				
13002802	0 0. 0.	1.539	3				
13002803	0 0. 0.	1.515	4				
13002804	0 0. 0.	0.8	5				
13002805	0 0. 0.	0.6	6				
13002806	0 0. 0.	0.4	7				

* UPPER SHELL

13003000	6 4 3 1	0.9					
13003100	0 2						
13003101	0.01 1	0.02 2	0.034 3				
13003201	5 3						
13003301	0. 3						
13003401	617.91 4						
13003501	320020000	0 1 1	0.18888 .1				
13003502	320030000	0 1 1	0.11111 2				
13003503	320040000	0 1 1	0.08333 3				
13003504	320050000	0 1 1	0.05555 4				
13003505	320060000	0 1 1	0.05555 5				
13003506	330010000	0 1 1	0.02777 6				
13003601	950010000	0	3950 1	0.18888	1		
13003602	950010000	0	3950 1	0.11111	2		
13003603	950010000	0	3950 1	0.08333	3		
13003604	950010000	0	3950 1	0.05555	4		
13003605	950010000	0	3950 1	0.05555	5		
13003606	950010000	0	3950 1	0.02777	6		
13003701	0 0. 0.	0. 6					
13003801	0 0. 0.	0.3 1					
13003802	0 0. 0.	0.2 2					
13003803	0 0. 0.	0.15 3					
13003804	0 0. 0.	0.1 4					
13003805	0 0. 0.	0.1 5					
13003806	0 0. 0.	0.05 6					

* STEAM GENERATOR

* COOLANT CHANNEL

```

11211000 1 7 1 1 0.
12211000 1 7 1 1 0.
11211100 0 2
12211100 1211
11211101 0.01 1 0.05 2 0.083 4 0.05 5 0.01 6
11211201 5 6
11211301 0. 6
11211501 121010000 0 1 0 7.1 1
11211601 125010000 0 1 0 7.1 1
11211701 0 0. 0. 0. 1
11211801 0 0. 0. 1.503 1
11211901 0 0. 0. 1.503 1
12211501 221010000 0 1 0 7.1 1
12211601 225010000 0 1 0 7.1 1
12211701 0 0. 0. 0. 1
12211801 0 0. 0. 1.503 1
12211901 0 0. 0. 1.503 1

```

* TUBE PLATE

```

*
11212000 2 2 1 1 0.
12212000 2 2 1 1 0.
12212100 1212
11212100 0 2
11212101 0.03056 1
11212201 5 1
11212301 0. 1
11212501 121010000 4000000 1 0 61. 2
11212601 620010000 0 1 0 61. 2
11212701 0 0. 0. 0. 2
11212801 0 0.01966 0.01966 .575 2
11212901 0 0. 0. 0. 2

```

```

*
12212501 221010000 4000000 1 0 61. 2
12212601 720010000 0 1 0 61. 2
12212701 0 0. 0. 0. 2
12212801 0 0.01966 0.01966 0.575 2
12212901 0 0. 0. 0. 2

```

* LOWER SHELL

```

*
16151000 3 4 2 1 1.5465
17151000 3 4 2 1 1.5465
16151100 0 2
17151100 6151
16151101 0.01 1 0.02 2 0.04526 3
16151201 5 3
16151301 0. 3
16151501 615040000 -10000 1 1 2.55 3
16151601 960010000 0 3960 1 2.55 3
16151701 0 0. 0. 0. 3
16151801 0 0. 0. 2.55 3
17151501 715040000 -10000 1 1 2.55 3
17151601 970010000 0 3960 1 2.55 3
17151701 0 0. 0. 0. 3

```

```

17151801 0 0. 0. 2.55 3
*
* TRANSITION CONE
16152000 1 4 2 1 1.7822
17152000 1 4 2 1 1.7822
16152100 0 2
17152100 6152
16152101 0.01 1 0.03 2 0.055 3
16152201 5 3
16152301 0. 3
16152501 615010000 0 1 1 1.937 1
16152601 960010000 0 3960 1 1.937 1
16152701 0 0. 0. 0. 1
16152801 0 0. 0. 1.937 1
17152501 715010000 0 1 1 1.937 1
17152601 970010000 0 3960 1 1.937 1
17152701 0 0. 0. 0. 1
17152801 0 0. 0. 1.937 1
*
* UPPER SHELL
*
* UPPER SHELL
16153000 5 4 2 1 2.2446
17153000 5 4 2 1 2.2446
16153100 0 2
17153100 6153
16153101 0.01 1 0.03 2 0.051 3
16153201 5 3
16153301 0. 3
16153501 613010000 0 1 1 0.696 1
16153502 612010000 0 1 1 0.735 2
16153503 611010000 0 1 1 2.424 3
16153504 627010000 0 1 1 1.000 4
16153505 630010000 0 1 1 2.093 5
16153601 960010000 0 3960 1 0.696 1
16153602 960010000 0 3960 1 0.735 2
16153603 960010000 0 3960 1 2.424 3
16153604 960010000 0 3960 1 1.000 4
16153605 960010000 0 3960 1 2.093 5
16153701 0 0. 0. 0. 5
16153801 0 0. 0. 0.696 1
16153802 0 0. 0. 0.735 2
16153803 0 0. 0. 2.424 3
16153804 0 0. 0. 1.000 4
16153805 0 0. 0. 2.093 5
17153501 713010000 0 1 1 0.696 1
17153502 712010000 0 1 1 0.735 2
17153503 711010000 0 1 1 2.424 3
17153504 727010000 0 1 1 1.000 4
17153505 730010000 0 1 1 2.093 5
17153601 970010000 0 3960 1 0.696 1
17153602 970010000 0 3960 1 0.735 2
17153603 970010000 0 3960 1 2.424 3
17153604 970010000 0 3960 1 1.000 4
17153605 970010000 0 3960 1 2.093 5

```

```

17153701 0 0. 0. 0. 5
17153801 0 0. 0. 0.696 1
17153802 0 0. 0. 0.735 2
17153803 0 0. 0. 2.424 3
17153804 0 0. 0. 1.000 4
17153805 0 0. 0. 2.093 5
*
* WRAPPER
16154000 8 2 2 1 1.524
17154000 8 2 2 1 1.524
16154100 0 2
17154100 6154
16154101 0.01 1
16154201 5 1
16154301 0. 1
16154501 620010000 10000 1 1 1.461 1
16154502 620020000 10000 1 1 1.816 4
16154503 621010000 0 1 1 2.324 5
16154504 622010000 0 1 1 0.696 6
16154505 622010000 0 1 1 0.735 7
16154506 623010000 0 1 1 2.424 8
16154601 615040000 -10000 1 1 1.461 1
16154602 615030000 -10000 1 1 1.816 4
16154604 615010000 0 1 1 2.324 5
16154605 613010000 0 1 1 0.696 6
16154606 612010000 0 1 1 0.735 7
16154607 611010000 0 1 1 2.424 8
16154701 0 0. 0. 0. 8
16154801 0 0. 0. 1.461 1
16154802 0 0. 0. 1.816 4
16154803 0 0. 0. 2.324 5
16154804 0 0. 0. 0.696 6
16154805 0 0. 0. 0.735 7
16154806 0 0. 0. 2.424 8
16154901 0 0. 0. 1.461 1
16154902 0 0. 0. 1.816 4
16154903 0 0. 0. 2.324 5
16154904 0 0. 0. 0.696 6
16154905 0 0. 0. 0.735 7
16154906 0 0. 0. 2.424 8
17154501 720010000 10000 1 1 1.461 1
17154502 720020000 10000 1 1 1.816 4
17154503 721010000 0 1 1 2.324 5
17154504 722010000 0 1 1 0.696 6
17154505 722010000 0 1 1 0.735 7
17154506 723010000 0 1 1 2.424 8
17154601 715040000 -10000 1 1 1.461 1
17154602 715030000 -10000 1 1 1.816 4
17154604 715010000 0 1 1 2.324 5
17154605 713010000 0 1 1 0.696 6
17154606 712010000 0 1 1 0.735 7
17154607 711010000 0 1 1 2.424 8
17154701 0 0. 0. 0. 8
17154801 0 0. 0. 1.461 1
17154802 0 0. 0. 1.816 4

```

17154803 0 0. 0. 2.324 5
 17154804 0 0. 0. 0.696 6
 17154805 0 0. 0. 0.735 7
 17154806 0 0. 0. 2.424 8
 17154901 0 0. 0. 1.461 1
 17154902 0 0. 0. 1.816 4
 17154903 0 0. 0. 2.324 5
 17154904 0 0. 0. 0.696 6
 17154905 0 0. 0. 0.735 7
 17154906 0 0. 0. 2.424 8

* INTERNAL STRUCTURES

*
 16203000 9 3 2 1 2.214
 17203000 9 3 2 1 2.214
 16203100 0 2
 17203100 6203
 16203101 0.015 2
 16203201 5 2
 16203301 0. 2
 16203501 620010000 10000 1 1 1.816 4
 16203503 621010000 0 1 1 2.324 5
 16203504 622010000 0 1 1 1.432 6
 16203506 623010000 0 1 1 2.424 7
 16203507 625010000 0 1 1 1.000 8
 16203508 630010000 0 1 1 2.093 9
 16203601 620010000 10000 1 1 1.816 4
 16203603 621010000 0 1 1 2.324 5
 16203604 622010000 0 1 1 1.432 6
 16203606 623010000 0 1 1 2.424 7
 16203607 625010000 0 1 1 1.000 8
 16203608 630010000 0 1 1 2.093 9
 16203701 0 0. 0. 0. 9
 16203801 0 0. 0. 1.816 4
 16203802 0 0. 0. 2.324 5
 16203803 0 0. 0. 1.432 6
 16203804 0 0. 0. 2.424 7
 16203805 0 0. 0. 1.000 8
 16203806 0 0. 0. 2.093 9
 16203901 0 0. 0. 1.816 4
 16203902 0 0. 0. 2.324 5
 16203903 0 0. 0. 1.432 6
 16203904 0 0. 0. 2.424 7
 16203905 0 0. 0. 1.000 8
 16203906 0 0. 0. 2.093 9
 17203501 720010000 10000 1 1 1.816 4
 17203503 721010000 0 1 1 2.324 5
 17203504 722010000 0 1 1 1.432 6
 17203505 723010000 0 1 1 2.424 7
 17203506 725010000 0 1 1 1.000 8
 17203507 730010000 0 1 1 2.093 9
 17203601 720010000 10000 1 1 1.816 4
 17203603 721010000 0 1 1 2.324 5
 17203604 722010000 0 1 1 1.432 6
 17203605 723010000 0 1 1 2.424 7

17203606	725010000					0	1	1	1.000	8
17203607	730010000					0	1	1	2.093	9
17203701	0	0.	0.	0.		9				
17203801	0	0.	0.	1.816		4				
17203802	0	0.	0.	2.324		5				
17203803	0	0.	0.	1.432		6				
17203805	0	0.	0.	2.424		7				
17203806	0	0.	0.	1.000		8				
17203807	0	0.	0.	2.093		9				
17203901	0	0.	0.	1.816		4				
17203902	0	0.	0.	2.324		5				
17203903	0	0.	0.	1.432		6				
17203905	0	0.	0.	2.424		7				
17203906	0	0.	0.	1.000		8				
17203907	0	0.	0.	2.093		9				

*
*

* STEAM LINES

*

16400000	3	3	2	1	0.30296					
17400000	3	3	2	1	0.30227					
16400100	0	2								
17400100	0	2								
16400101	0.01804			1	0.02	2				
17400101	0.02			1	0.02346	2				
16400201	3	2								
17400201	3	2								
16400301	0.	2								
17400301	0.	2								
16400501	640010000		10000	1	1	22.717	3			
17400501	740010000		10000	1	1	16.242	3			
16400601		0	0	0	1	22.717	3			
17400601		0	0	0	1	16.242	3			
16400701	0	0	0.	0.	3					
17400701	0	0.	0.	0.	3					
16400801	0	0.	0.	0.	3					
17400801	0	0	0.	0.	3					

* VESSEL

*

* COVER

*

10451000	1	6	3	1	1.705						
10451100	0	2									
10451101	0.005		1	0.005	2	0.01	3	0.100	4	0.216	5
10451201	3	1									
10451202	5	5									
10451301	0										
10451501	0450.0000		0	1	1	0.5	1				
10451601		0	0	0	1	0.5	1				
10451701	0	0.	0	0.	1						
10451801	0	0.	0.	1.496	1						

*

*WALL 045-043

```

*
10452000 1 5 1 0 0.
10452100 0 2
10452101 0.01 1 0.04 3 0.01 4
10452201 3 4
10452301 0. 4
10452501 045010000 0 1 0 20.51 1
10452601 043010000 0 1 0 20.51 1
10452701 0 0. 0. 0. 1
10452801 0 0. 0. 0. 1
10452901 0 0. 0. 0. 1

```

* BOTTOM

```

*
10210000 2 5 3 1 1.695
10210100 0 2
10210101 0.005 1 0.025 2 0.03 3 0.055 4
10210201 3 1
10210202 5 4
10210301 0. 4
10210501 021010000 2000000 1 1 0.25 2
10210601 0 0 0 0 1 0.25 2
10210701 0 0. 0. 0. 2
10210801 0 0. 0. 0.638 2

```

* DOWNCOMER WALL

```

*
10150000 3 20 2 1 1.6635
10150100 0 2
10150101 1.25E-3 5 2.00E-3 6 3.00E-3 7 4.00E-3 8 5.0E-3 9 6.00E-3 10
10150102 8.00E-3 11 10.E-3 12 12.E-3 13 15.E-3 14 20.E-3 16
10150103 23.916E-3 19
10150201 3 4
10150202 5 19
10150301 0. 19
10150501 015010000 10000 1 1 1.361 2
10150502 015030000 1000 1 1 1.362 3
10150601 0 0 0 1 1.361 2
10150602 0 0 0 1 1.362 3
10150701 0 0. 0. 0. 3
10150801 0 0. 0. 1.361 2
10150802 0 0. 0. 1.362 3

```

* UPPER PLENUM

```

*
10110000 2 7 2 1 1.6635
10110100 0 2
10110101 0.005 2 0.03 3 0.06 4 0.10 5 0.137 6
10110201 3 1
10110202 5 6
10110301 0. 6
10110501 011010000 0 1 1 1.584 1
10110502 013010000 0 1 1 0.872 2
10110601 0 0 0 1 1.584 1
10110602 0 0 0 1 0.872 2

```

10110701 0 0. 0. 0. 2
 10110801 0 0. 0. 1.584 1
 10110802 0 0. 0. 0.872 2

* THERMAL SHIELD

*
 10151000 3 3 2 1 1.4635
 10151100 0 2
 10151101 0.04525 2
 10151201 3 2
 10151301 0. 2
 10151501 015010000 10000 1 1 1.136 3
 10151601 015010000 10000 1 1 1.136 3
 10151701 0 0. 0. 0. 3
 10151801 0 0. 0. 1.136 3
 10151901 0 0. 0. 1.136 3

* INTERNAL STRUCTURES

*
 10300000 10 3 2 1 6.255
 10300100 0 2
 10300101 0.01 2
 10300201 3 2
 10300301 0. 2
 10300501 021010000 2000000 1 1 0.638 2
 10300502 025010000 0 1 1 1.013 3
 10300503 031010000 10000 1 1 0.6095 7
 10300504 039010000 0 1 1 0.6095 8
 10300505 041010000 0 1 1 0.8965 9
 10300506 043010000 0 1 1 1.584 10
 10300601 021010000 2000000 1 1 0.638 2
 10300602 025010000 0 1 1 1.013 3
 10300603 031010000 10000 1 1 0.6095 7
 10300604 039010000 0 1 1 0.6095 8
 10300605 041010000 0 1 1 0.8965 9
 10300606 043010000 0 1 1 1.584 10
 10300701 0 0. 0. 0. 10
 10300801 0 0. 0. 0.638 2
 10300802 0 0. 0. 1.013 3
 10300803 0 0. 0. 0.6095 8
 10300804 0 0. 0. 1. 9
 10300805 0 0. 0. 1.584 10
 10300901 0 0. 0. 0.638 2
 10300902 0 0. 0. 1.013 3
 10300903 0 0. 0. 0.6095 8
 10300904 0 0. 0. 0.8965 9
 10300905 0 0. 0. 1.584 10

* PRIMARY LEGS

* HOT-COLD LEGS

11110000 3 3 2 1 0.34925
 11510000 4 3 2 1 0.34925
 12110000 3 3 2 1 0.34925
 12510000 4 3 2 1 0.34925

```

11110100 0 2
11510100 1110
12110100 1110
12510100 1110
11110101 0.031375 2
11110201 3 2
11110301 0. 2
11110501 111010000 1000000 1 1 2.476 2
11110502 113010000 0 1 1 2.476 3
12110503 211010000 10000 1 1 2.476 3
11510504 151010000 10000 1 1 1.968 2
11510505 152010000 1000000 1 1 1.968 4
12510506 251010000 10000 1 1 1.968 2
12510507 252010000 1000000 1 1 1.968 4
11110601 0 0 0 0 1 2.476 3
11510602 0 0 0 0 1 1.968 4
11110701 0 0. 0. 0. 3
11510701 0 0. 0. 0. 4
11110801 0 0. 0. 0.6985 3
11510801 0 0. 0. 0.6985 4
12110601 0 0 0 1 2.478 3
12510601 0 0 0 1 1.968 4
12110701 0 0. 0. 0. 3
12510701 0 0. 0. 0. 4
12110801 0 0. 0. 0.6985 3
12510801 0 0. 0. 0.6985 4
*
* PUMP SUCTION LEG
*
11310000 4 3 2 1 0.3937
11310100 0 2
12310000 4 3 2 1 0.3937
12310100 1310
11310101 0.035125 2
11310201 3 2
11310301 0. 2
11310501 131010000 10000 1 1 1.806 1
11310502 132010000 10000 1 1 1.806 2
12310502 231010000 10000 1 1 1.806 2
11310503 133010000 0 1 1 1.633 3
11310504 133020000 0 1 1 1.43 4
11310601 0 0 0 1 1.806 2
11310602 0 0 0 1 1.633 3
11310603 0 0 0 1 1.43 4
11310701 0 0. 0. 0. 4
11310801 0 0. 0. 1.806 2
11310802 0 0. 0. 0.7874 3
11310803 0. 0. 0. 1.43 4
12310503 231030000 0 1 1 1.633 3
12310504 231040000 0 1 1 1.430 4
12310601 0 0 0 1 1.806 2
12310602 0 0 0 1 1.633 3
12310603 0 0 0 1 1.430 4
12310701 0 0. 0. 0. 4
12310801 0 0. 0. 1.806 2

```

```

12310802 0 0. 0. 0.7874 3
12310803 0 0. 0. 1.43 4
*
* PUMP
*
11400000 1 5 3 1 1.228
11400100 0 2
12400000 1 5 3 1 1.228
12400100 1400
11400101 0.01 1 0.03 2 0.08 3 0.162 4
11400201 5 4
11400301 0. 4
11400501 140010000 0 1 1 1. 1
11400601 0 0 0 1 1. 1
11400701 0 0. 0. 0. 1
11400801 0 0. 0. 1.232 1
12400501 240010000 0 1 1 1. 1
12400601 0 0 0 1 1. 1
12400701 0 0. 0. 0. 1
12400801 0 0. 0. 1.232 1

```

```

*****
* INITIAL CONDITIONS
*****

```

* MASS FLOW RATE

```

*
0111201 16.900 0. 0.
0112201 16.900 0. 0.
0131201 7239.1 0. 0.
0132201 3628. 0. 0.
0133201 3628. 0. 0.
0151301 7239.1 0. 0. 2
0231201 7239.1 0. 0.
0232201 7239.1 0. 0.
0233201 0. 0. 0.
0251201 6911.8 0. 0.
0252201 327.33 0. 0.
0311301 6912. 0. 0. 3
0391201 6912. 0. 0.
0392201 327.35 0. 0.
0393201 7239.1 0. 0.
0431201 7239.1 0. 0.
0432201 3628. 0. 0.
0433201 3628. 0. 0.
0451201 -16.894 0. 0.
1121201 3628. 0. 0.
1122201 3628. 0. 0.
1123201 0. 0. 0.
1211201 3628. 0. 0.
1212201 3628. 0. 0.
1221301 3628. 0. 0. 9
1231201 3628. 0. 0.
1232201 3628. 0. 0.
1241301 3628. 0. 0. 4

```

1251201	3628.	0.	0.	
1252201	3628.	0.	0.	
1311201	3628.	0.	0.	
1321201	3628.	0.	0.	
1331301	3628.	0.	0.	1
1400201	1 3628.	0.	0.	
1400202	1 3628.	0.	0.	
1511301	3628.	0.	0.	1
1521201	3628.	0.	0.	
1522201	3628.	0.	0.	
2111301	3628.	0.	0.	2
2211201	3628.	0.	0.	
2212201	3628.	0.	0.	
2221301	3628.	0.	0.	9
2230201	1 3628.	0.	0.	
2241301	3628.	0.	0.	4
2251201	3628.	0.	0.	
2252201	3628.	0.	0.	
2311301	3628.	0.	0.	3
2400201	1 3628.	0.	0.	
2400202	1 3628.	0.	0.	
2511301	3628.	0.	0.	1
2521201	3628.	0.	0.	
2522201	3628.	0.	0.	
3011301	0.	0.	0.	2
3050201	1	0.	0.	0.

*

6111201	0.	0.	0.		
6112201	0.	0.	0.		
6121201	0.	0.	0.	0.	
6301201	0.00	0.	0.		
6140201	0	.0		.0	
6151301	0.	0.	0.	.0	1
6151302	0.	0.	0.	.0	2
6151303	0.	0.	0.	.0	3
6170201	0	0.	0.	.0	
6201301	.0	0.	0.	0.	3
6211201	0.	0.	0.		
6231201	0.	0.	0.		
6232201	0.	0.	0.		
6233201	0.	0.	0.		
6212201	0.	0.	0.		
6401301	0.	0.	0.	0.	2
7111201	0.	0.	0.		
7112201	0.	0.	0.		
7121201	0.	0.	0.	0.	
7301201	0.00	0.	0.		
7140201	0	.0		.0	
7151301	0.	0.	0.	.0	1
7151302	0.	0.	0.	.0	2
7151303	0.	0.	0.	.0	3
7170201	0	0.	0.	.0	
7201301	.0	0.	0.	0.	1
7201302	.0	0.	0.	0.	2

7201303	.0	0.	0.	3
7211201	.0	0.	0.	
7212201	.0	0.	0.	
7231201	0.	0.	0.	
7232201	0.	0.	0.	
7233201	0.	0.	0.	
7401301	0.	0.	0.	2
3011301	0.	0.	0.	2
3050201	1	0.	0.	0.

*
* HEAT STRUCTURES TEMPERATURES

10110401	528.	7
10150401	528.	20
10151401	528.	3
10210401	528.	5
10300400	0	
10300401	528.	3
10451401	528.	6
10452401	528.	5
11110401	528.	3
11510401	528.	3
11211401	528.	7
11212401	528.	2
12212401	528.	2
11310401	528.	3
11400401	528.	5
13000401	528.	3
16151401	528.	4
16152401	528.	4
16153401	528.	4
16154401	528.	2
16203401	528.	3
16202401	528.	2
16400401	528.	3
17400401	528.	3
17202401	528.	2
12110401	528.	3
12510401	528.	3
12211401	528.	7
12310401	528.	3
12400401	528.	5
17151401	528.	4
17152401	528.	4
17153401	528.	4
17154401	528.	2
17203401	528.	3

* PRESSURE, INTERNAL ENERGY, QUALITY

0110200	3	155.E5	528.				
0130200	3	155.E5	528.				
0151201	3	155.E5	528.	0.	0.	0.	3
0210200	3	155.E5	528.				
0230200	3	155.E5	528.				
0250200	3	155.E5	528.				

0311201	3	155.E5	528.	0.	0.	0. 4
0350200	3	155.E5	528.			
0390200	3	155.E5	528.			
0410200	3	155.E5	528.			
0430200	3	155.E5	528.			
0450200	3	155.E5	528.			
1110200	3	155.E5	528.			
1120200	3	155.E5	528.			
1131202	3	155.E5	528.	0.	0.	0. 2
1210200	3	155.E5	528.			
1221201	3	155.E5	528.	0.	0.	0. 8
1230200	3	155.E5	528.			
1241201	3	155.E5	528.	0.	0.	0. 5
1250200	3	155.E5	528.			
1311201	3	155.E5	528.	0.	0.	0. 4
1400200	3	155.E5	528.			
1511201	3	155.E5	528.	0.	0.	0. 2
1520200	3	155.E5	528.			
1530200	3	155.E5	528.			
2111201	3	155.E5	528.	0.	0.	0. 4
2210200	3	155.E5	528.			
2221201	3	155.E5	528.	0.	0.	0. 10
2241201	3	155.E5	528.	0.	0.	0. 5
2250200	3	155.E5	528.			
2311201	3	155.E5	528.	0.	0.	0. 4
2400200	3	155.E5	528.			
2511201	3	155.E5	528.	0.	0.	0. 2
2520200	3	155.E5	528.			
2530200	3	155.E5	528.			
3011201	2	155.E5	0.	0.	0.	0. 3
6110200	1	528.	0.2277			
6120200	1	528.	0.			
6130200	1	528.	0.			
6151201	1	528.	0.	0.	0.	0. 4
6201201	1	528.	0.	0.	0.	0. 4
6210200	1	528.	0.			
6220200	1	528.	0.			
6300200	1	528.	1.			
6230200	1	528.	.2277			
6401201	1	528.	1.	0.	0.	0. 3
7110200	1	528.	0.2277			
7120200	1	528.	0.			
7130200	1	528.	0.			
7151201	1	528.	0.	0.	0.	0. 4
7201201	1	528.	0.	0.	0.	0. 4
7210200	1	528.	0.			
7220200	1	528.	0.			
7230200	1	528.	0.2277			
7300200	1	528.	1.0000			
7401201	1	528.	1.	0.	0.	0. 3

*
*

* THERMAL PROPERTIES *

```

*
*      N 1   U02
*      2   ZR
*      3   S-STEEL
*      4   GAP
*      5   C-STEEL
*      6   INCONEL-600
*
*****

```

```

*
20100100 TBL/FCTN 1 1 * U02
20100101 293. 8.37
20100102 366. 6.577
20100103 533. 5.78
20100104 616. 4.623
20100105 700. 4.633
20100106 783. 4.221
20100107 866. 3.88
20100108 950. 3.596
20100109 1033. 3.357
20100110 1200. 2.984
20100111 1977. 2.29
20100112 3088. 2.99
*

```

```

20100151 273. 2.31E6
20100152 323. 2.57E6
20100153 373. 2.75E6
20100154 473. 2.92E6
20100155 673. 3.13E6
20100156 1373. 3.44E6
20100157 4700. 6.80E6
*

```

```

*
20100200 TBL/FCTN 1 1 * ZR
20100201 273.15 7.
20100202 473.15 1.200438E1
20100203 673.15 1.400510E1
20100204 873.15 1.700793E1
20100205 1073.15 1.900866E1
20100206 1273.15 2.200975E1
20100207 1473.15 2.501085E1
20100208 1673.15 3.001267E1
20100209 1873.15 3.601486E1
20100210 2073.15 4.401777E1
20100211 2273.15 5.502352E1
20100212 2473.15 6.802826E1
*

```

```

20100251 255.3722 1.904141E6
20100252 1077.594 2.312171E6
20100253 1185.928 5.712422E6
20100254 1248.428 2.311769E6
20100255 2199.817 2.312171E6
*

```

```

20100300 TBL/FCTN 1 1 *S-STEEL
20100301 273. 13.
20100302 1190. 25.

```

```

*
20100351 225. 3.5E6
20100352 580. 4.34E6
20100353 1300. 5.29E6
*
20100400 TBL/FCTN 1 1 *GAP
20100401 0.56 *REF.:STABILIZATION VESSEL.
20100451 5.4 *RHO-CD EGG-LOFT-5480
*

```

```

20100500 TBL/FCTN 1 1 *C-STEEL
20100501 46.05
20100551 3.8775E6
*

```

```

20100600 TBL/FCTN 1 1 *INCONNEL-600
20100601 294.15 14.86
20100602 366.15 15.76
20100603 477.15 17.45
20100604 589.15 19.18
20100605 700.15 20.91
*

```

```

20100651 294.15 3.88E6
20100652 366.15 3.88E6
20100653 477.15 4.227E6
20100654 589.15 4.227E6
20100655 700.15 4.580E6
*

```

```

*****
*CONTROL SYSTEM
*****

```

```

*****
* SATURATION TEMPERATURE *
*****

```

```

9990000 TV999 TMDPVOL
9990101 10. 1. 0. 0 0. 0. 0. 0. 00
9990200 2 0 P 113010000
9990201 -1. 155.E5 0.
9990202 1.E5 1.E5 0.
9990203 200.E5 200.E5 0.
*

```

```

20500100 BREAK INTEGRAL 1. 0. 0
20500101 MFLOWJ 618000000
*

```

```

*****
* MASS BALANCE *
*****

```

```

20500200 MASSBIL SUM 1. 0. 0
20500201 0. -1. CNTRLVAR 001 *BREAK
20500202 1. CNTRLVAR 079 *SI
20500203 1. CNTRLVAR 180 *CV
20500206 -1. CNTRLVAR 004 *LETDOWN
*

```

```

20500400 LETDO INTEGRAL 1. 0. 0

```

```

20500401 CNTRLVAR 188
*
20500500 LETENERG. MULT 1. 0. 1
20500501 CNTRLVAR 188 CNTRLVAR 093
*
20500600 CVENERGY MULT 1. 0. 1
20500601 MFLOWJ 181000000 CNTRLVAR 083
*
20500700 BREAKENER MULT 1. 0. 1
20500701 CNTRLVAR 012 CNTRLVAR 081
*
20500900 ENERGY SUM 1. 0. 1
20500901 0. 1. CNTRLVAR 031 * CORE POWER
20500902 -1. CNTRLVAR 610 * SGB TRANSFER
20500903 -1. CNTRLVAR 710 * SGA TRANSFER
20500904 73190. CNTRLVAR 078 * SI
20500905 1. CNTRLVAR 006 * CV
20500906 -1. CNTRLVAR 007 * BREAK
20500907 1. CNTRLVAR 993 * METAL
20500909 -1. CNTRLVAR 005 * LETD
20500911 1. CNTRLVAR 140 * PUMP B
20500912 1. CNTRLVAR 240 * PUMP A
*
20501000 SUMW SUM 1. 0. 1
20501001 0. 1. CNTRLVAR 078 * SI
20501002 1. MFLOWJ 181000000 * CV
20501003 -1. MFLOWJ 618000000 * BREAK
20501006 -1. CNTRLVAR 188 * LETDOWN
*
20501200 BRKDIFF DIFFREND. 1. 0.0 0
20501201 CNTRLVAR 001
*
20501300 CVLET SUM 1.0 0. 0
20501301 0. 1. CNTRLVAR 006
20501302 -1. CNTRLVAR 005
*
20503100 COREPUW SUM 1. 6.E6 0
20503101 0. 1. 0 031010000
20503102 1. 0 031020000
20503103 1. 0 031030000
20503104 1 0 031040000
*
20503300 LEVELCORE SUM 1. 0. 1
20503301 0 0.638 VOIDF 021010000
20503302 0.638 VOIDF 023010000
20503303 1.013 VOIDF 025010000
20503304 0.6095 VOIDF 031010000
20503305 0.6095 VOIDF 031020000

```

* ENERGY *

* ENERGY BALANCE *

* ADDITION RATE *

** CORE POWER *

*** CORE COLLAPSED LEVEL **

20503306 0.6095 VOIDF 031030000
 20503307 0.6095 VOIDF 031040000
 20503308 0.6095 VOIDF 039010000
 20503309 0.8965 VOIDF 041010000

*
 20503400 HEADLEV SUM 1. 0. 1
 20503401 O. 1.584 VOIDF 043010000
 20503402 1.496 VOIDF 045010000

 ** UPPER HEAD COLLAPSED LEVEL *

*
 20503500 DOWNLEV SUM 1. 0. 1
 20503501 O. 0.638 VOIDF 021010000
 20503502 0.638 VOIDF 023010000
 20503503 1.362 VOIDF 015030000
 20503504 1.361 VOIDF 015020000
 20503505 1.361 VOIDF 015010000
 20503506 0.872 VOIDF 013010000

 ** DOWNCOMER COLLAPSED LEVEL **

*
 *
 20504000 VESSMASS SUM 1. 0. 1
 20504001 O. 3.4357 RHO 011010000
 20504002 1.8202 RHO 013010000
 20504003 1.9190 RHO 015010000
 20504004 1.9190 RHO 015020000
 20504005 1.9190 RHO 015030000
 20504006 2.6790 RHO 021010000
 20504007 5.3590 RHO 023010000
 20504008 3.5740 RHO 025010000
 20504009 1.5238 RHO 031010000
 20504010 1.5238 RHO 031020000
 20504011 1.5238 RHO 031030000
 20504012 1.5238 RHO 031040000
 20504013 1.9706 RHO 035010000
 20504014 2.3252 RHO 039010000
 20504015 5.1030 RHO 041010000
 20504016 9.3490 RHO 043010000
 20504017 8.8330 RHO 045010000

 ** VESSEL MASS INVENTORY *

*
 *
 20504100 LOOPB SUM 1. 0. 1
 20504101 O. .9483 RHO 111010000
 20504102 .9483 RHO 112010000
 20504103 .9483 RHO 113010000
 20504104 .87952 RHO 131010000
 20504105 .87952 RHO 132010000
 20504106 .79527 RHO 133010000
 20504107 .69641 RHO 133020000
 20504108 .75374 RHO 151010000
 20504109 .75374 RHO 152010000
 20504110 .75374 RHO 153010000
 20504113 .75374 RHO 151020000
 20504111 7.7570 RHO 140010000

 ** LOOP B MASS INVENTORY *

```

*
*
20504200 LOOPA SUM 1. 0. 1
20504201 O. .9483 RHO 211010000
20504202 .9483 RHO 211020000
20504203 .9483 RHO 211030000
20504204 .87952 RHO 231010000
20504205 .87952 RHO 231020000
20504206 .79527 RHO 231030000
20504207 .69641 RHO 231040000
20504208 .75374 RHO 251010000
20504213 .75374 RHO 251020000
20504209 .75374 RHO 252010000
20504210 .75374 RHO 253010000
20504211 7.7570 RHO 240010000
*

```

```

*****
** LOOP A MASS INVENTORY *
*****

```

```

*
*
20504300 SGB SUM 1. 0. 1
20504301 O. .89892 RHO 122010000
20504302 .89892 RHO 122020000
20504303 .89892 RHO 122030000
20504304 .89892 RHO 122040000
20504305 .89892 RHO 122050000
20504306 .89892 RHO 122060000
20504307 .89892 RHO 122070000
20504308 .89892 RHO 122080000
20504310 1.7978 RHO 123010000
20504311 1.7978 RHO 124010000
20504312 1.7978 RHO 124020000
20504313 1.7978 RHO 124030000
20504314 1.7978 RHO 124040000
20504315 1.7978 RHO 124050000
20504316 4.3920 RHO 121010000
20504317 4.3920 RHO 125010000
*

```

```

*****
** UTUBE SGA MASS INVENTORY *
*****

```

```

*
*
20504400 SGA SUM 1. 0. 1
20504401 O. .89892 RHO 222010000
20504402 .89892 RHO 222020000
20504403 .89892 RHO 222030000
20504404 .89892 RHO 222040000
20504405 .89892 RHO 222050000
20504406 .89892 RHO 222060000
20504407 .89892 RHO 222070000
20504408 .89892 RHO 222080000
20504409 .89892 RHO 222090000
20504410 .89892 RHO 222100000
20504411 1.7978 RHO 224010000
20504412 1.7978 RHO 224020000
20504413 1.7978 RHO 224030000
20504414 1.7978 RHO 224040000
20504415 1.7978 RHO 224050000
20504416 4.3920 RHO 221010000
20504417 4.3920 RHO 225010000
*

```

```

*****
** UTUBE SGA MASS INVENTORY *
*****

```

```

*****

```

```

20504500 PRES SUM 1. 0. 1
20504501 O. .24593 RHO 301010000
20504502 .24593 RHO 301020000
20504503 .15771 RHO 301030000
20504504 .50892 RHO 310010000
20504505 1.0178 RHO 311010000
20504506 4.4440 RHO 312010000
20504507 4.4440 RHO 313010000
20504508 3.8727 RHO 314010000
20504509 3.8123 RHO 315010000
20504510 2.0131 RHO 316010000
20504511 1.5098 RHO 317010000
20504512 1.0066 RHO 320010000
20504513 .50892 RHO 320020000
20504514 .33928 RHO 320030000
20504515 .25446 RHO 320040000
20504516 .16964 RHO 320050000
20504517 .16964 RHO 320060000
20504518 8.482E-2 RHO 330010000

```

```

** PRESU MASS INVENTORY *
*****

```

```

*
*
20505000 PRIMASS SUM 8.5982E-6 O. 1
20505001 O. 1. CNTRLVAR 045
20505002 1. CNTRLVAR 044
20505003 1. CNTRLVAR 043
20505004 1. CNTRLVAR 042
20505005 1. CNTRLVAR 041
20505006 1. CNTRLVAR 040

```

```

*****
* PRIMARY MASS *
*****

```

```

*
*
20507000 PO11 SUM 1. 107.910E5 O
20507001 O. 0.1 P 043010000
20507002 0.9 CNTRLVAR 070

```

```

*****
** SI SYSTEM *
*****

```

```

*
20507100 SISUM SUM 1. 0. 0
20507101 O. 1. MFLOWJ 171000000
20507102 1. MFLOWJ 271000000

```

```

*
20507800 SISUM SUM 1. 0. 1
20507802 O. 1. CNTRLVAR 071

```

```

*
20507900 SIBILAN INTEGRAL 1. 0. 0
20507901 CNTRLVAR 078

```

```

*
20508000 P/RHO122 DIV 1. 0. 1
20508001 RHOF 122080000 P 122080000

```

```

*
20508100 ENTHA122 SUM 1. 0. 1
20508101 O. 1. UF 122080000
20508102 1. CNTRLVAR 080

```

```

*
20508200 P/RHO180 DIV 1. 0. 1
20508201 RHOF 180010000 P 180010000

```

```

20508300 ENTHA180 SUM 1 0. 1
20508301 O. 1. UF 180010000
20508302 1. CNTRLVAR 082
*
*20508400 P/RHOG DIV 1. 0. 1
*20508401 RHOG 777010000 P 777010000
*
*20508500 P/RHOF DIV 1. 0. 1
*20508501 RHOF 777010000 P 777010000
*
*20508600 ENTHAF?? SUM 1. 0. 1
*20508601 O. 1. UF 777010000
*20508602 1. CNTRLVAR 085
*
*
*20508700 ENTHAG?? SUM 1. 0. 1
*20508701 O. 1. UG 777010000
*20508702 1. CNTRLVAR 084
*
*20508800 XENTHAG?? MULT 1. 0. 1
*20508801 CNTRLVAR 087 QUALE 777010000
*
*20508900 XENTHAL?? MULT 1. 0. 1
*20508901 CNTRLVAR 086 QUALE 777010000
*
*20509000 ENTHA?? SUM 1. 0. 1
*20509001 O. 1. CNTRLVAR 086
*20509002 1. CNTRLVAR 088
*20509003 -1. CNTRLVAR 089
*
20509200 P/RHO131 DIV 1 G. 1
20509201 RHOF 131010000 P 131010000
*
20509300 ENTHA131 SUM 1. 0. 1
20509301 O. 1. UF 131010000
20509302 1. CNTRLVAR 092
*
*
20511300 TAVG SUM 0.25 575.33 1
20511301 O. 1. TEMPF 113010000
20511302 1. TEMPF 151010000
20511303 1. TEMPF 251020000
20511304 1. TEMPF 211030000
*
20511400 SUBCOOLING SUM 1. 0. 1
20511401 O. 1. TEMPF 999010000
20511402 -1. TEMPF 113010000
*
*
*
20512200 LEVCSGB SUM 0.908 0. 1
20512201 O. 1. VOIDF 122010000
20512202 1. VOIDF 122020000

```

```

*****
**AVERAGED TEMPERATURE*
*****

```

```

*****
* SGB U TUBE LEVEL *
*****

```

20512203	1.	VOIDF	122030000
20512204	1.	VOIDF	122040000
20512205	1.	VOIDF	122050000
20512206	1.	VOIDF	122060000
20512207	1.	VOIDF	122070000
20512208	1.	VOIDF	122080000
20512209	1.414	VOIDF	123010000

20512400	LEVHQB	SUM	1.816	0.	1
20512401	0.	0.707	VOIDF	124010000	
20512402	1.		VOIDF	124020000	
20512403	1.		VOIDF	124030000	
20512404	1.		VOIDF	124040000	
20512405	1.		VOIDF	124050000	

 ** LOOP SEAL B COLLAPSED LEVEL *

20513100	SEALB	SUM	1.	0.	1
20513101	0.	1.806	VOIDF	131010000	
20513102		1.806	VOIDF	132010000	

20513200	SEALB	SUM	1.	0.	1
20513201	0.	1.43	VOIDF	133020000	

20514000 PUMPB TRIPUNIT 2.44E8 2.44E8 0
 20514001 602

 ** CHARGING SYSTEM *

20518000 INTCV INTEGRAL 1. 0. 0
 20518001 MFLOWJ 181000000

20518800 LETDOWN SUM 1. 0. 0
 20518801 0. 1. MFLOWJ 183000000

 * SGA U TUBE LEVEL *

20522200	LEVCSGA	SUM	0.908	0.	1
20522201	0.	1.	VOIDF	222010000	
20522202	1.		VOIDF	222020000	
20522203	1.		VOIDF	222030000	
20522204	1.		VOIDF	222040000	
20522205	1.		VOIDF	222050000	
20522206	1.		VOIDF	222060000	
20522207	1.		VOIDF	222070000	
20522208	1.		VOIDF	222080000	
20522209	0.707		VOIDF	222090000	
20522210	0.707		VOIDF	222100000	

```

*
20522400 LEVHSGA SUM 1.816 O. 1
20522401 O. 0.707 VOIDF 224010000
20522402 1. VOIDF 224020000
20522403 1. VOIDF 224030000
20522404 1. VOIDF 224040000
20522405 1. VOIDF 224050000

```

```

*
*
*
*

```

```

*****
** LOOP SEAL A COLLAPSED LEVEL **
*****

```

```

20523100 SEALA SUM 1. O. 1
20523101 O. 1.806 VOIDF 231010000
20523102 1.806 VOIDF 231020000
*
20523200 SEALA SUM 1. O. 1
20523201 O. 1.43 VOIDF 231040000

```

```

*
*

```

```

20524000 PUMPA TRIPUNIT 2.44E6 2.44E6 0
20524001 598

```

```

*
*

```

```

20542000 FRBL SUM 9.81 O. 1
20542001 O. 0.908 RHO 122010000
20542002 0.908 RHO 122020000
20542003 0.908 RHO 122030000
20542004 0.908 RHO 122040000
20542005 0.908 RHO 122050000
20542006 0.908 RHO 122060000
20542007 0.908 RHO 122070000
20542008 0.908 RHO 122080000
20542009 1.284 RHO 123010000
20542011 1.685 RHO 121010000
20542012 0.950 RHO 113010000
20542013 1.430 RHO 133020000
20542014 1.232 RHO 140010000

```

```

*****
** DRIVING FORCE IN LOOP B **
*****

```

```

20542100 FMBL SUM 9.81 O. 1
20542101 O. 1.284 RHO 124010000
20542102 1.816 RHO 124020000
20542103 1.816 RHO 124030000
20542104 1.816 RHO 124040000
20542105 1.816 RHO 124050000
20542106 1.685 RHO 125010000
20542107 1.806 RHO 131010000
20542108 1.806 RHO 132010000

```

```

20542200 FMFRBL SUM 1.0 O. 1
20542201 O. 1. CNTRLVAR 421
20542202 -1. CNTRLVAR 420

```

```
*
20542300 FMFRTOBL SUM 1.0 0. 1
20542301 0. 1. CNTRLVAR 422
20542302 1. 1. CNTRLVAR 400
*
```

```
*
20541000 FRAL SUM 9.81 0. 1
20541001 0. 0.908 RHO 222010000
20541002 0.908 RHO 222020000
20541003 0.908 RHO 222030000
20541004 0.908 RHO 222040000
20541005 0.908 RHO 222050000
20541006 0.908 RHO 222060000
20541007 0.908 RHO 222070000
20541008 0.908 RHO 222080000
20541009 0.642 RHO 222090000
20541010 0.642 RHO 222100000
20541011 1.685 RHO 221010000
20541012 0.950 RHO 211030000
20541013 1.430 RHO 231040000
20541014 1.232 RHO 240010000
*
```

```
*****
** DRIVING FORCE IN LOOP A *
*****
```

```
*
20541100 FMAL SUM 9.81 0. 1
20541101 0. 1.284 RHO 224010000
20541102 1.816 RHO 224020000
20541103 1.816 RHO 224030000
20541104 1.816 RHO 224040000
20541105 1.816 RHO 224050000
20541106 1.685 RHO 225010000
20541107 1.806 RHO 231010000
20541108 1.806 RHO 231020000
*
```

```
*
20541200 FMFRAL SUM 1.0 0. 1
20541201 0. 1. CNTRLVAR 411
20541202 -1. CNTRLVAR 410
*
```

```
*
20541300 FMFRTOAL SUM 1.0 0. 1
20541301 0. 1. CNTRLVAR 412
20541302 1. CNTRLVAR 400
*
```

```
*
20540000 FMFRCORE SUM 9.81 0. 1
20540001 0. 0.872 RHO 013010000
*
20540003 1.361 RHO 015010000
20540004 1.361 RHO 015020000
20540005 1.362 RHO 015030000
20540006 -1.013 RHO 025010000
20540007 -0.6095 RHO 031010000
20540008 -0.6095 RHO 031020000
20540009 -0.6095 RHO 031030000
20540010 -0.6095 RHO 031040000
20540011 -0.6095 RHO 039010000
20540012 -0.8965 RHO 041010000
*
```

```
*****
** DRIVING FORCE IN VESSEL *
*****
```

```

*
*
20560000 SGBMASS SUM 1. 0. 1
20560001 0. 19.508 RHO 611010000
20560002 5.596 RHO 613010000
20560003 5.015 RHO 615010000
20560004 1.2355 RHO 615020000
20560005 1.2355 RHO 615030000
20560006 1.2355 RHO 615040000
20560007 8.049 RHO 620010000
20560008 8.049 RHO 620020000
20560009 8.049 RHO 620030000
20560010 8.049 RHO 620040000
20560011 10.3 RHO 621010000
20560014 5.728 RHO 622010000
20560015 9.696 RHO 623010000
20560016 26.288 RHO 630010000
20560017 4.000 RHO 625010000
20560018 8.560 RHO 627010000
20560012 5.90940 RHO 612010000
*
*

```

```

*****
** SG B MASS INVENTORY *
*****

```

```

*
*
20560300 G600 SUM 1. 0. 1
20560301 0. 1. MFLOWJ 618000000
20560302 1. MFLOWJ 607000000
20560303 1. MFLOWJ 608000000
20560309 -1. MFLOWJ 865000000
*

```

```

*****
* SGB ADDITION RATES *
*****

```

```

20560400 BILSGB INTEGRAL 1. 0. 1
20560401 CNTRLVAR 603
*

```

```

*
*
20560900 SGBPOWPR SUM 1. 5.96E8 0
20560901 0. 1. Q 121010000
20560902 1. Q 122010000
20560903 1. Q 122020000
20560904 1. Q 122030000
20560905 1. Q 122040000
20560906 1. Q 122050000
20560907 1. Q 122060000
20560908 1. Q 122070000
20560909 1. Q 122080000
20560910 1. Q 123010000
20560911 1. Q 124010000
20560912 1. Q 124020000
20560913 1. Q 124030000
20560914 1. Q 124040000
20560915 1. Q 124050000
20560916 1. Q 125010000
*

```

```

*****
** SGB HEAT TRANSFER RATE *
*****

```

```

*
*
20561000 SGBPOW SUM 1. 5.96E8 0
20561001 0. 1. Q 620010000

```

20561002 1. Q 620020000
 20561003 1. Q 620030000
 20561004 1. Q 620040000
 20561005 1. Q 621010000

*
 *
 *

20561100 LSGB SUM 1. 11.662 1
 20561101 O. 2.424 VOIDF 611010000
 20561102 0.696 VOIDF 613010000
 20561103 1.937 VOIDF 615010000
 20561104 2.55 VOIDF 615020000
 20561105 2.55 VOIDF 615030000
 20561106 2.55 VOIDF 615040000
 20561107 0.735 VOIDF 612010000
 20561108 2.093 VOIDF 630010000
 20561109 1.000 VOIDF 627010000

*
 *
 *

20561200 LSGBRI SUM 1. 5.71 1
 20561201 O. 1.816 VOIDF 620010000 * CHECK?
 20561202 1.816 VOIDF 620020000 *
 20561203 1.816 VOIDF 620030000 *
 20561204 1.816 VOIDF 620040000 *
 20561205 2.324 VOIDF 621010000 *
 20561206 1.432 VOIDF 622010000 *
 20561207 2.424 VOIDF 623010000

*
 *
 *
 *

20561300 DPLSGB SUM 3.794E-5 G. 1 3 0 1
 20561301 -1068.9 4.6303 RHO 613010000
 20561302 23.779 RHO 611010000
 20561303 7.2104 RHO 612010000

*
 *
 *
 *

20570000 SGBMASS SUM 1. 0. 1
 20570001 O. 19.508 RHO 711010000
 20570002 5.596 RHO 713010000
 20570003 5.015 RHO 715010000
 20570004 1.2355 RHO 715020000
 20570005 1.2355 RHO 715030000
 20570006 1.2355 RHO 715040000
 20570007 8.049 RHO 720010000
 20570008 8.049 RHO 720020000
 20570009 8.049 RHO 720030000
 20570010 8.049 RHO 720040000
 20570011 10.3 RHO 721010000
 20570014 5.728 RHO 722010000
 20570015 9.696 RHO 723010000

 ** LEVEL SGB DOWNCOMER *

 ** LEVEL SGB RISER *

 ** SGB LEVEL DP *

 ** SG A MASS INVENTORY *

20570016 26.288 RHO 730010000
 20570017 4.000 RHO 725010000
 20570018 8.560 RHO 727010000
 20570012 5.90940 RHO 712010000

*
 *
 *

20570300 G700 SUM 1. 0. 1
 20570301 0. 1. MFLOWJ 707000000
 20570303 1. MFLOWJ 708000000
 20570310 -1. MFLOWJ 791000000

 * ADDITION RATES *

*
 *

20570400 INTG7 INTEGRAL 1. 0. 1
 20570401 CNTRLVAR 703

*
 *
 *

20570900 SGAPOWPR SUM 1. 5.96E8 0

 ** SGA HEAT TRANSFER RATE *

20570901 0. 1. Q 221010000
 20570902 1. Q 222010000
 20570903 1. Q 222020000
 20570904 1. Q 222030000
 20570905 1. Q 222040000
 20570906 1. Q 222050000
 20570907 1. Q 222060000
 20570908 1. Q 222070000
 20570909 1. Q 222080000
 20570910 1. Q 222090000
 20570911 1. Q 222100000
 20570912 1. Q 224010000
 20570913 1. Q 224020000
 20570914 1. Q 224030000
 20570915 1. Q 224040000
 20570916 1. Q 224050000
 20570917 1. Q 225010000

*
 *
 *

20571000 SGAPOW SUM 1. 5.96E8 0

20571001 0. 1. Q 720010000
 20571002 1. Q 720020000
 20571003 1. Q 720030000
 20571004 1. Q 720040000
 20571005 1. Q 721010000

*
 *
 *

20571100 LSGA SUM 1. 11.662 1
 20571101 0. 2.424 VOIDF 711010000
 20571102 0.696 VOIDF 713010000
 20571103 1.937 VOIDF 715010000
 20571104 2.55 VOIDF 715020000

 ** LEVEL SGA DOWNCOMER *

20571105 2.55 VOIDF 715030000
 20571106 2.55 VOIDF 715040000
 20571107 0.735 VOIDF 712010000
 20571108 2.093 VOIDF 730010000
 20571109 1.000 VOIDF 727010000

*
 *

20571200 LSGARI SUM 1. 5.7 1
 20571201 0. 1.816 VOIDF 720010000
 20571202 1.816 VOIDF 720020000
 20571203 1.816 VOIDF 720030000
 20571204 1.816 VOIDF 720040000
 20571205 2.324 VOIDF 721010000
 20571206 1.432 VOIDF 722010000
 20571207 2.424 VOIDF 723010000

*
 *

20571300 DPLSGA SUM 3.794E-5 0. 1 3 0. 1.
 20571301 -1068.9 4.6303 RHO 713010000
 20571302 23.779 RHO 711010000
 20571303 7.2104 RHO 712010000

*
 *

*
 *

20590000 LE1PRESS SUM 0.048209 0. 1
 20590001 -2.906086 4.44396 VOIDF 312010000
 20590002 4.44396 VOIDF 313010000
 20590003 3.873 VOIDF 314010000
 20590004 3.81208 VOIDF 315010000
 20590005 2.01312 VOIDF 316010000
 20590006 1.50984 VOIDF 317010000
 20590007 1.00656 VOIDF 320010000
 20590008 5.0892E-1 VOIDF 320020000
 20590009 3.3928E-1 VOIDF 320030000
 20590010 2.5446E-1 VOIDF 320040000
 20590011 1.6964E-1 VOIDF 320050000
 20590012 1.6964E-1 VOIDF 320060000
 20590013 8.4620E-2 VOIDF 330010000
 20590014 1.0178 VOIDF 311010000
 20590015 0.24593 VOIDF 301010000
 20590016 0.24593 VOIDF 301020000
 20590017 0.15771 VOIDF 301030000
 20590018 0.50892 VOIDF 310010000

*
 *

20590100 LE2PRES SUM 2.515E-5 0. 1 3 0
 20590101 -8248. 14.4796 RHO 312010000
 20590102 17.324 RHO 313010000
 20590103 15.0976 RHO 314010000
 20590104 14.862 RHO 315010000
 20590105 7.848 RHO 316010000
 20590106 5.886 RHO 317010000
 20590107 3.924 RHO 320010000
 20590108 1.4224 RHO 320020000

 ** LEVEL RISER SGA

 ** SGA LEVEL DP *

 ** PRESSURIZER LEVEL *

```

*
20590200 LE3PRES SUM 1.2459E-5 O. 1 3 O. 1.
20590201 O. 14.4796 RHO 312010000
20590202 17.324 RHO 313010000
20590203 15.0976 RHO 314010000
20590204 14.862 RHO 315010000
20590205 7.848 RHO 316010000
20590206 5.886 RHO 317010000
20590207 3.924 RHO 320010000
20590208 1.4224 RHO 320020000

```

```

*****
* SG B FW + AFW *
*****

```

```

20596900 (AFW+FW) SUM 1. O. 1
20596902 O. 1. MFLOWJ 607000000
20596903 1. MFLOWJ 608000000

```

```

*****
* SG A FW + AFW *
*****

```

```

20597900 (AFW+FW) SUM 1. O. 1
20597902 O. 1. MFLOWJ 707000000
20597903 1. MFLOWJ 708000000

```

```

*****
* HEAT LOSSES *
*****

```

```

20598900 HEATLOSGA SUM 1. O. 1
20598901 O. 1. Q 970010000
*
20599000 HEATLOSGB SUM 1. O. 1
20599001 O. 1. Q 960010000
*
20599100 HEATLOSPR SUM 1. O. 1
20599101 O. 1. Q 950010000

```

```

*****
* WALLS HEAT TRANSFER *
*****

```

```

20599200 QPRIM1 SUM 1. O. 1
20599201 O. 1. Q 011010000
20599202 1. Q 013010000
20599203 1. Q 015010000
20599204 1. Q 015020000
20599205 1. Q 015030000
20599206 1. Q 021010000
20599207 1. Q 023010000
20599208 1. Q 025010000
20599209 1. Q 035010000
20599210 1. Q 039010000
20599211 1. Q 041010000
20599212 1. Q 043010000

```

```

** VESSEL **

```

20599213 1. Q 045010000

*
*

20599300 QLOOPB SUM 1. O. 1

** LOOP B **

20599301 0. 1. Q 132010000

20599302 1. Q 133010000

20599303 1. Q 133020000

20599304 1. Q 140010000

20599305 1. Q 151010000

20599306 1. Q 151020000

20599307 1. Q 152010000

20599308 1. Q 153010000

20599309 1. Q 111010000

20599310 1. Q 112010000

20599311 1. Q 113010000

20599312 1. Q 131010000

*
*

*
*

20599400 QPRESS SUM 1. O. 1

** PRESSURISER**

20599401 0. 1. Q 330010000

20599402 1. Q 310010000

20599404 1. Q 301010000

20599405 1. Q 301020000

20599406 1. Q 301030000

20599407 1. Q 311010000

20599408 1. Q 312010000

20599409 1. Q 313010000

20599410 1. Q 314010000

20599411 1. Q 315010000

20599412 1. Q 316010000

20599413 1. Q 317010000

20599414 1. Q 320010000

20599415 1. Q 320020000

20599416 1. Q 320030000

20599417 1. Q 320040000

20599418 1. Q 320050000

20599419 1. Q 320060000

*
*

*
*

20599500 QLOOPA SUM 1. O. 1

** LOOP A **

20599501 0. 1. Q 251020000

20599502 1. Q 252010000

20599503 1. Q 253010000

20599504 1. Q 211010000

20599505 1. Q 211020000

20599506 1. Q 211030000

20599507 1. Q 231010000

20599508 1. Q 231020000

20599509 1. Q 231030000

20599510 1. Q 231040000

20599511 1. Q 240010000

20599512 1. Q 251010000

*
*

```

20599600 QPRINT SUM 1. 0. 1
20599601 0. 1. CNTRLVAR 993
20599602 1. CNTRLVAR 995
20599603 1. CNTRLVAR 994
20599604 1. CNTRLVAR 992

```

** TOTAL **

*
*
*
*

```

*****
*TURBO PUMP MODEL
*****

```

```

6080000 J608 TMDPJUN
6080101 606000000 613000000 1.
6080200 1 0 CNTRLVAR 870
6080201 -1. 0. 0. 0.
6080202 0. 0. 0. 0.
6080203 50. 50. 0. 0.

```

```

7080000 J708 TMDPJUN
7080101 706000000 713000000 1.
7080200 1 0 CNTRLVAR 871
7080201 -1. 0. 0. 0.
7080202 0. 0. 0. 0.
7080203 50. 50. 0. 0.

```

```

20586200 TRIPB TRIPUNIT 1. 1. 0
20586201 661

```

```

20586300 TRIPA TRIPUNIT 1. 1. 0
20586301 661

```

```

20586400 P-PB SUM 1. 0. 0
20586401 0. 1. CNTRLVAR 882
20586402 -1. P 613010000

```

```

20586500 P-PA SUM 1. 0. 0
20586501 0. 1. CNTRLVAR 882
20586502 -1. P 713010000

```

```

20586600 WPB2 SUM 7.22E-5 0. 0
20586601 0. 1. CNTRLVAR 864
20586602 -181. CNTRLVAR 873

```

```

20586700 WPA2 SUM 7.22E-5 0. 0
20586701 0. 1. CNTRLVAR 865
20586702 -181. CNTRLVAR 873

```

```

20586800 WPB POWERR 1. 0. 0
20586801 CNTRLVAR 866 0.5

```

```

20586900 WPA POWERR 1. 0. 0
20586901 CNTRLVAR 867 0.5

```

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```

20587000 WPB MULT 1. 0. 1
20587001 CNTRLVAR 868 CNTRLVAR 862
*
20587100 WPA MULT 1. 0. 1
20587101 CNTRLVAR 869 CNTRLVAR 863
*
20587200 WPA+WPB SUM 0.5 0. 0.
20587201 0. 1. CNTRLVAR 870
20587202 1. CNTRLVAR 871
20587203 1. CNTRLVAR 872
*
20587300 WP2 POWERI 1. 0. 0
20587301 CNTRLVAR 872 2
*
20587400 EQ18 MULT 4.734 0. 0 1
20587401 CNTRLVAR 872 CNTRLVAR 878
*
20587500 VP SUM 0.5 368. 0
20587501 0. 244. CNTRLVAR 876
20587502 1. CNTRLVAR 874
20587503 1. CNTRLVAR 875
*
20587600 VT/VP DIV 368. 1. 0 2 1.
20587601 CNTRLVAR 875
*
20587700 N/NO POWERR 1. 1. 0 3 0.5 1.
20587701 CNTRLVAR 876 0.33333
*
20587800 (N/NO)2 POWERI 1. 1. 0 2 1.
20587801 CNTRLVAR 877 2
*
20588000 EQ17 SUM 1. 0. 0
20588001 0. 1. CNTRLVAR 872
20588002 -8.89 CNTRLVAR 877
*
20588100 EQ17 POWERI 1. 1. 0
20588101 CNTRLVAR 880 2
*
20588200 PPUMP SUM 0.5 92.4E5 0 2 92.4E5
20588201 0. 1. CNTRLVAR 882
20588202 -1918. CNTRLVAR 881
20588203 92.4E5 CNTRLVAR 878
*
20588300 WT SUM 1. 0. 0 2 2.344
20588301 0.75 0.00433 CNTRLVAR 875

```

```

*****
*GENERAL TABLES
*****
*
*
*POWER TABLE
*
```

```

20210000 POWER
20210001 O. 6.E6
*
* PRESSURIZER HEATERS
20231100 POWER 642
20231101 -1. O.
20231102 O. 850.E3
*
*HEAT TRANSFER COEFF. PRESSU-CONTAINEMENT
*
20295000 HTC-T
20295001 O. 1.91
*
*HEAT TRANSFER COEFF. SG-CONTAINEMENT
*
20296000 HTC-T
20296001 O. 0.95
*
*
*****
** PLOT REQUEST
*****
20300010 DOTGRID
*
*
20302100 P 330010000
*
20303100 CNTRLVAR 900 901 902
20303200 CNTRLVAR 901
*
20304100 P 211020000
*
20305200 TEMPF 015030000 041010000
20305400 TEMPF 211030000
20305500 TEMPF 113010000
20305600 TEMPF 251020000
20305700 TEMPF 151020000
20305800 CNTRLVAR 113
20305900 TEMPG 330010000
*
20306000 CNTRLVAR 002
20306100 CNTRLVAR 012
20306200 MFLOWJ 618000000
20306300 CNTRLVAR 010
20306400 CNTRLVAR 079
20306500 MFLOWJ 181000000 183000000
20306600 CNTRLVAR 012 078
*
20307100 CNTRLVAR 031
20307200 CNTRLVAR 007
20307300 CNTRLVAR 710 610
20307500 CNTRLVAR 009

```

```

20307600 CNTRLVAR 996 994
20307700 CNTRLVAR 031 007
*
20308100 MFLOWJ 013020000 013030000
20308300 MFLOWJ 301010000
20308400 PMPVEL 140 240
20308500 MFLOWJ 345000000 3460000000
*
20309200 CNTRLVAR 114
20309300 P 015030000
20309310 TEMPF
20309900 CNTRLVAR 050
*
20310100 P 730010000 630010000
20310200 P 730010000
20310300 P 630010000
*
20311100 CNTRLVAR 711 611 712 612
20311500 CNTRLVAR 713 613
20311600 CNTRLVAR 711
20311700 CNTRLVAR 611
20311800 CNTRLVAR 713
20311900 CNTRLVAR 613
*
20312100 TEMPG 630010000 730010000
20312200 TEMPF 121010000 620040000 125010000
20312300 TEMPF 121010000 720040000 225010000
20312400 TEMPF 620010000 621010000 623010000 630010000
*
20314700 MFLOWJ 865000000
20314800 CNTRLVAR 979 989
20314900 MFLOWJ 791000000
*
20315100 CNTRLVAR 600 700
20315200 CNTRLVAR 603 703
*
20318300 CNTRLVAR 001
20318400 TEMPF 043010000 999010000
*
20320000 CPUTIME 0

```

```

*****
* TITLES
*****

```

```

20302130 "FIG 2.1 PRESSURE EVOLUTION IN PRESSURIZER "
*
20303130 " FIG 3.1 WATER LEVEL IN PRESSURIZER "
20303230 " FIG 3.2 MEASURED WATER LEVEL IN PRESSURIZER "
*
20304130 " FIG 4.1 PRESSURE EVOLUTION IN HOT LEG A"
*
20305230 "FIG 5.2 TEMPERATURE EVOLUTION IN REACTOR VESSEL "
20305430 " FIG 5.4 TEMPERATURE EVOLUTION IN HOT LEG A"

```

20305530 " FIG 5.5 TEMPERATURE EVOLUTION IN HOT LEG B"
20305630 " FIG 5.6 TEMPERATURE EVOLUTION IN COLD LEG A"
20305730 " FIG 5.7 TEMPERATURE EVOLUTION IN COLD LEG B"
20305830 " FIG 5.8 AVERAGED PRIMARY TEMPERATURE EVOLUTION "
20305930 " FIG 5.9 PRESSURIZER STEAM TEMPERATURE EVOLUTION "
*
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20306130 " FIG 6.1 BREAK MASS FLOW RATE (CORRECTED)"
20306230 " FIG 6.2 BREAK MASS FLOW RATE "
20306330 " FIG 6.3 PRIMARY SYSTEM MASS ADDITION AND DEPLETION RATES "
20306430 " FIG 6.4 INTEGRAL OF HPSI MASS FLOW RATE"
20306530 " FIG 6.5 CHARGING AND LETDOWN MASS FLOW RATE "
20306630 " FIG 6.6 BREAK AND HPSI MASS FLOW RATE "
*
20307130 " FIG 7.1 CORE THERMAL POWER "
20307230 " FIG 7.2 BREAK ENERGY EVOLUTION "
20307330 " FIG 7.3 HEAT TRANSFER RATE FROM PRIMARY TO SG "
20307530 " FIG 7.5 PRIMARY SYSTEM ENERGY BALANCE "
20307630 " FIG 7.6 HEAT ADDITION RATE FROM PRIMARY METAL STRUCTURES "
20307730 " FIG 7.7 CORE AND BREAK POWER "
*
20308130 " FIG 8.1 COLD LEG MASS FLOW RATE (VESSEL INLET)"
20308330 " FIG 8.3 SURGELINE MASS FLOW RATE "
20308430 " FIG 8.4 PUMP VELOCITY"
20308530 " FIG 8.5 SPRAY MASS FLOW RATE "
*
20309230 " FIG 9.2 SUBCOOLING IN PRIMARY SYSTEM "
20309330 " FIG 9.3 PRESSURE VERSUS TEMPERATURE IN DOWNCOMER"
20309930 " FIG 9.9 MASS BALANCE IN PRIMARY SYSTEM"
*
20310130 " FIG 10.1 PRESSURE EVOLUTION IN STEAM GENERATOR (DOME) "
20310230 " FIG 10.2 PRESSURE EVOLUTION IN STEAM GENERATOR A(DOME) "
20310330 " FIG 10.3 PRESSURE EVOLUTION IN STEAM GENERATOR B(DOME) "
*
20311130 " FIG 11.1 COLLAPSED WATER LEVEL IN STEAM GENERATORS "
20311530 " FIG 11.5 MEASURED WATER LEVEL IN STEAM GENERATORS "
20311630 " FIG 11.6 COLLAPSED WATER LEVEL IN STEAM GENERATOR A "
20311730 " FIG 11.7 COLLAPSED WATER LEVEL IN STEAM GENERATOR B "
20311830 " FIG 11.8 MEASURED WATER LEVEL IN STEAM GENERATOR A "
20311930 " FIG 11.9 MEASURED WATER LEVEL IN STEAM GENERATOR B "
*
20312130 " FIG 12.1 STEAM TEMPERATURE IN STEAM GENERATORS(DOME) "
20312230 " FIG 12.2 LOOP B AND SG B TEMPERATURES "
20312330 " FIG 12.3 LOOP A AND SG A TEMPERATURES "
20312430 " FIG 12.4 TEMPERATURE STRATIFICATION IN SG B "
*
20314730 " FIG 14.7 STEAM MASS FLOW RATE FROM SGB TO TURBOPUMP "
20314830 " FIG 14.8 FW AND AFW TO STEAM GENERATOR "
20314930 " FIG 14.9 SG A RELIEF VALVE MASS FLOW RATE "
*
20315130 " FIG 15.1 STEAM GENERATORS MASS EVOLUTION "
20315230 " FIG 15.2 SG'S MASS ADDITION AND DEPLETION RATES "
*
20318330 " FIG 9.12 INTEGRAL OF BREAK MASS FLOW RATE "
20318430 " FIG 18.4 SUBCOOLING "

```

*
*
20320030 " FIG 20 CPUTIME VERSUS TRANSIENT TIME "
*****
* LINE AND CURVE SPECIFICATIONS *
*****
*
20303150 "COLLAPSED" "LEVEL" LINE 1 40
20303151 "HOT CELL" "LEVEL" LINE 2 40
20303152 "COLD CELL" "LEVEL" LINE 3 40
*
20303250 "HOT CELL" "LEVEL" LINE 1 40
*
20304150 " P " "21102" LINE 1 40
*
20305250 "DOWNC." "TEMP" LINE 1 40
20305251 "CORE OUTL." "TEMP" LINE 2 40
*
20305450 "LOOP " "A" LINE 1 40
*
20305550 "LOOP " "B" LINE 1 40
*
20305650 "LOOP " "A" LINE 1 40
*
20305750 "LOOP " "B" LINE 1 40
*
20306550 "MFLOW " "CV" LINE 1 40
20306551 "MFLOW " "LETD." LINE 2 40
*
20306650 "BREAK " "FLOW" LINE 1 40
20306651 "HPSI " "FLOW" LINE 2 40
*
*
20307350 "SG " "A" LINE 1 40
20307351 "SG " "B" LINE 2 40
*
20307650 "PRIMARY" "SYST" LINE 1 40
20307651 "PRESSU " "SYST" LINE 2 40
*
20307750 "CORE " "POWER" LINE 1 40
20307751 "BREAK " "POWER" LINE 2 40
*
20308150 "LOOP " "B" LINE 1 40
20308151 "LOOP " "A" LINE 2 40
*
20308450 "LOOP " "B" LINE 1 40
20308451 "LOOP " "A" LINE 2 40
*
20308550 "LOOP " "B" LINE 1 40
20308551 "LOOP " "A" LINE 2 40
*
*
20310150 " P " "SG A" LINE 1 35
20310151 " P " "SG B" LINE 1 40
*

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```

20310250 " P " "SG A" LINE 1 35
*
20310350 " P " "SG B" LINE 1 40
*
20311150 "DOWNC " "SG A" LINE 1 40
20311151 "DOWNC " "SG B" LINE 2 40
20311152 "RISER " "SG A" LINE 3 40
20311153 "RISER " "SG B" LINE 4 40
*
20311550 "DOWNC " "SGA" LINE 1 40
20311551 "DOWNC " "SGB" LINE 2 40
*
20311650 "LEVEL " "711" LINE 1 40
20311750 "LEVEL " "611" LINE 1 40
20311850 "LEVEL " "713" LINE 1 40
20311950 "LEVEL " "613" LINE 1 40
*
20312150 " SG " "B" LINE 1 40
20312151 " SG " "A" LINE 2 40
*
20312250 "INLET" "BOX" LINE 1 40
20312251 "RISER " "SG B" LINE 2 40
20312252 "OUTLET" "BOX" LINE 3 40
*
20312350 "INLET" "BOX" LINE 1 40
20312351 "RISER " "SG A" LINE 2 40
20312352 "OUTLET" "BOX" LINE 3 40
*
20312450 "TUBE BUNDLE" "BOT." LINE 1 40
20312451 "TUBE BUNDLE" "TOP" LINE 2 40
20312452 "SEPARATOR" " " LINE 3 40
20312453 "DOME" " " LINE 4 40
*
20314850 "SG " "A" LINE 1 40
20314851 "SG " "B" LINE 2 40
*
20315150 "SG " "B" LINE 1 40
20315151 "SG " "A" LINE 2 40
*
20315250 "SG " "B" LINE 1 40
20315251 "SG " "A" LINE 2 40
*
20318450 "CORE" "TEMP" LINE 1 40
20318451 "SAT." "TEMP" LINE 2 40

```

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*****
* AXES TITLES *
*****

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```

20303132 "LEVEL " "(%)"
20303232 "LEVEL " "(%)"
*
20305832 "TEMP " "(K)"
*

```

```

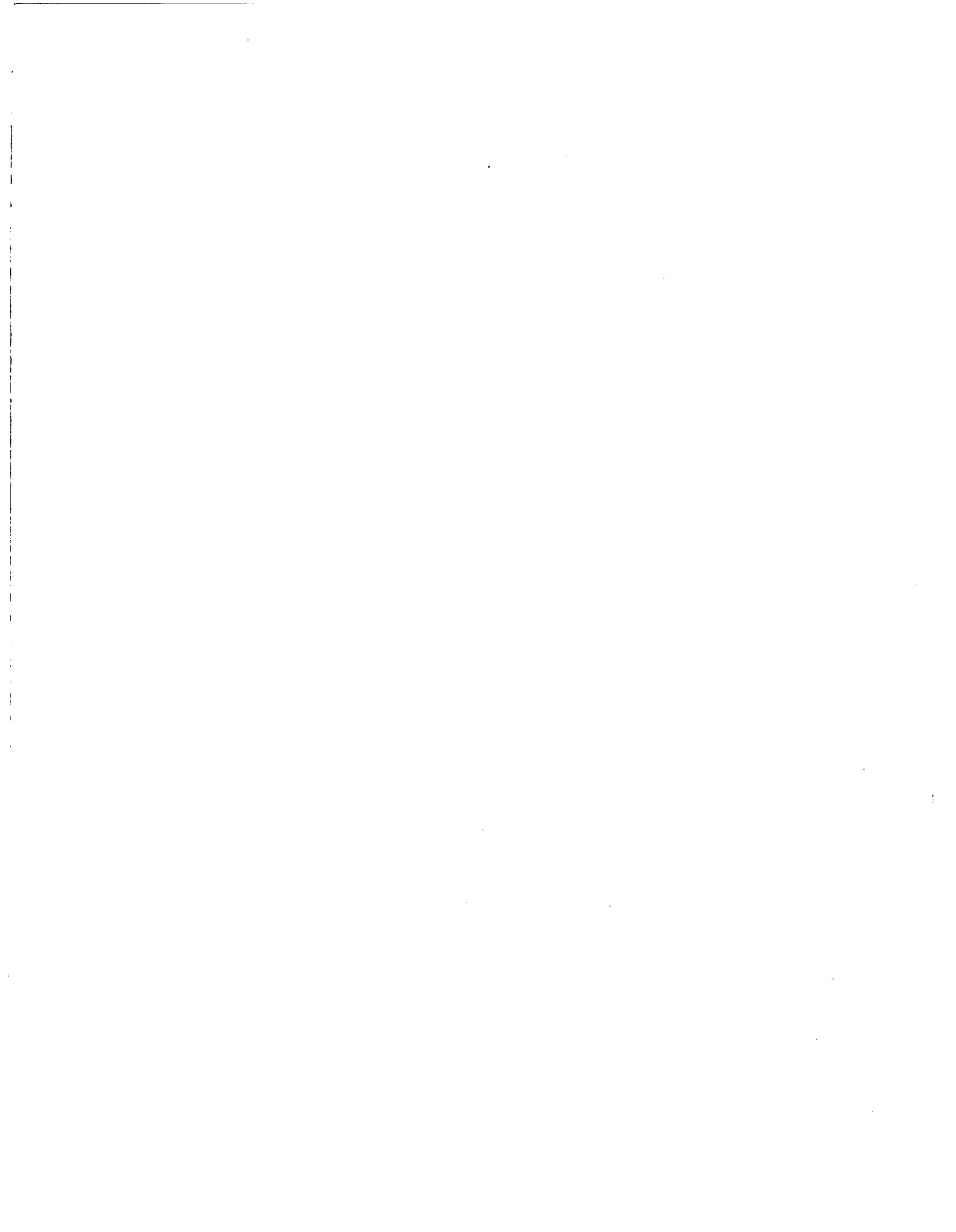
20306032 "MASS " "(KG)"
*
20306132 "MFLOW " "(KG/SEC)"
*
20306332 "MFLOW " "(KG/SEC)"
*
20306632 "MFLOW " "(KG/SEC)"
*
20307132 "POWER " "(WATT)"
*
20307232 "BREAK POWER" "(WATT)"
*
20307332 "POWER " "(WATT)"
*
20307532 "POWER " "(WATT)"
*
20307632 "POWER " "(WATT)"
*
20307732 "POWER " "(WATT)"
*
*
20309232 "DELTA TEMP" "(K)"
*
20311132 "LEVEL " "(M)"
*
20311532 "LEVEL " "(%)"
*
20314832 "MFLOW " "(KG/SEC)"
*
20315132 "MASS " "(KG)"
*
20318332 "MASS" "(KG)"
*
*
*****
* Y-AXE LIMITS *
*****
*
*****
* PLOTS COMPARISON *****
*
20304120 O4100 *FIG.14 PRESSURE EVOLUTION HOT LEG (LOOP-A)(PRA-RC 11)
20404100 P * 211020000
20404119 "PRA-RC" "11" CDOTS
20404120 O. 155.E05 152. 148.E05 422. 138.E05 828. 128.E05 980. 128.E05
20404121 1132. 108.E05 1470. 107.E05 2100. 107.E05 2281. 74.E05 2417. 99.E05
20404122 2636. 105.E05
*
20303120 O3100 *FIG.15 WATER LEVEL IN PRESSURIZER (LRCA/PR11)
20303220 O3100 *FIG.15 WATER LEVEL IN PRESSURIZER (LRCA/PR11)
20403100 CNTRLVAR
20403119 "LRCA-PR" "11" CDOTS
20403120 O.O 0.25
    
```

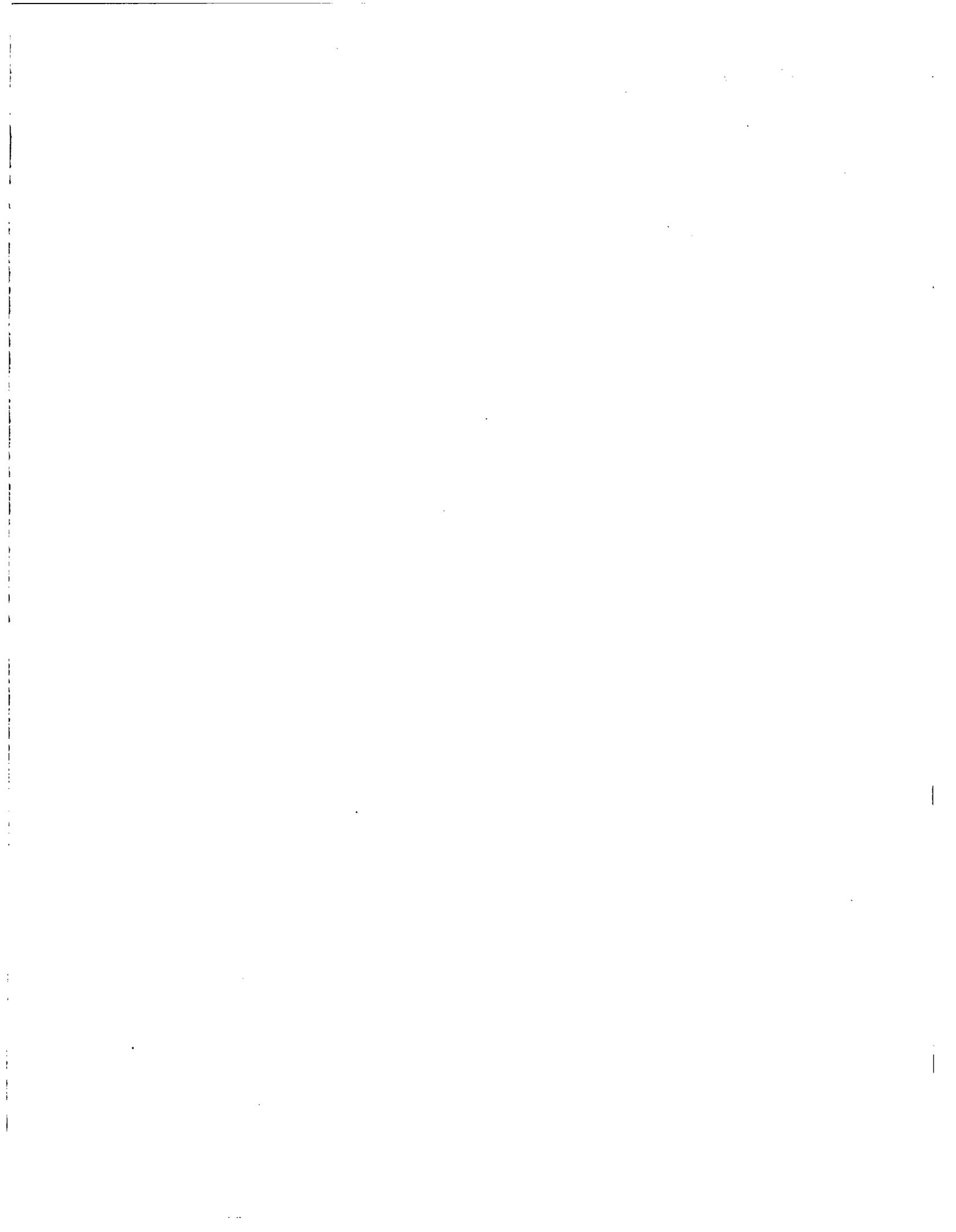
20403121 47.4 .197 284.2 .065 362. .032 442. 0. 2196. 0.
20403122 2368. 1. 2700. 1.
*
20312620 11200 *FIG.17 WATER LEVEL IN INTACT S.G. A (2 FW 9A)
20412600 CNTRLVAR
20412619 "2 FW " "9A" CDOTS
20412620 0. 11.217 871. 11.21 888. 11.187 890. 11.402 1006. 11.115
20412621 1008. 11.053 1142. 11.112 1260. 10.834 1429. 10.583 1716. 11.184
20412622 1817. 11.379 2021. 11.608 2190. 11.608 2527. 11.953
*
20311720 11300 *FIG.18 WATER LEVEL IN BROKEN S.G. B (2 FW 9B)
20411700 CNTRLVAR
20411719 "2 FW " "9B" CDOTS
20411720 0. 11.229 1484. 13.231 1535. 13.443 2700. 13.443
*
20311820 11500 *FIG.17 WATER LEVEL IN INTACT S.G. A (2 FW 9A)
20411800 CNTRLVAR
20411819 "2 FW " "9A" CDOTS
20411820 0. 0.39 871. 0.385 888. 0.379 890. 0.438 1006. 0.359
20411821 1008. 0.342 1142. 0.358 1260. 0.282 1429. 0.213 1716. 0.378
20411822 1817. 0.432 2021. 0.495 2190. 0.495 2527. 0.590
*
20311920 11600 *FIG.18 WATER LEVEL IN BROKEN S.G. B (2 FW 9B)
20411900 CNTRLVAR
20411919 "2 FW " "9B" CDOTS
20411920 0. 0.39 1484. 0.94 1535. 1.
*
20305620 05600 *FIG.19 TEMPERATURE IN COLD LEG A (2 RC 09)
20405600 TEMPF *251020000
20405619 "2 RC " "09" CDOTS *!!!!!!!!!!!!CHECK
20405620 0. 529. 763. 528. 1255. 524. 1565. 514. 1843. 502.
20405621 2155. 493. 2395. 482. 2816. 472.
*
20305420 05400 *FIG.20 TEMPERATURE IN COLD LEG A (2 RC 05)
20405400 TEMPF *211040000
20405419 "2 RC " "05" CDOTS *!!!!!!!!!!!!CHECK
20405420 0. 529. 763. 528. 1255. 524. 1565. 514. 1843. 502.
20405421 2155. 493. 2395. 482. 2816. 472.
*
20305720 05700 *FIG.21 TEMPERATURE IN COLD LEG B (2 RC 29)
20405700 TEMPF *151020000
20405719 "2 RC " "29" CDOTS *!!!!!!!!!!!!CHECK
20405720 0. 529. 763. 528. 1255. 524. 1565. 514. 1843. 502.
20405721 2155. 493. 2395. 482. 2816. 472.
*
20305520 05500 *FIG.22 TEMPERATURE IN HOT LEG B (2 RC 25)
20405500 TEMPF *211040000
20405519 "2 RC " "25" CDOTS *!!!!!!!!!!!!CHECK
20405520 0. 529. 763. 528. 1255. 524. 1565. 514. 1843. 502.
20405521 2155. 493. 2395. 482. 2816. 472.
*
20310120 10100 10200 *FIG.23 PRESSURE IN S.G. A (2 MS 04 A)
20410100 P * 734010000
20410119 "2 MS " "04A" CDOTS
20410120 0. 43.1E05 412. 44.4E05 1000. 45.2E05 1084. 43.4E05 1237 43.1E05

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20410121 1338. 40.6E05 1591. 35.8E05 1744. 30.7E05 1845. 29.E05 1946. 25.8E05
20410122 2183 23.5E05 2183. 22.2E05 2301. 20.8E05 2453. 18.4E05 3112. 14.6E05
*
20310220 10100
20310320 10200
*
20410200 P * SG B
20410219 "2 MS " "04B" DASHES
20410220 0. 42.85E05 149. 43.1E05 369. 44.1E05 673. 44.7E05 876. 45.E05
20410221 1197. 44.9E05 1484. 45.E05 1687. 40.4E05 1890. 35.63E05 2177. 37.95E05
20400222 2397 38.71E05 2598. 39.86E05
*
*
*
* END OF DATA
*
*
/*EOR
/*EOI

NRC FORM 335 (2-84) NRCM 1102, 3201, 3202 BIBLIOGRAPHIC DATA SHEET SEE INSTRUCTIONS ON THE REVERSE.		U.S. NUCLEAR REGULATORY COMMISSION		1. REPORT NUMBER (Assigned by TIDC, add Vol No., if any) NUREG/IA-0008	
2. TITLE AND SUBTITLE Assessment Study of RELAP-5 MOD-2 Cycle 36.01: Based on the Doel-2 Steam Generator Tube Rupture Incident of June 1979			3. LEAVE BLANK		
5. AUTHOR(S) E.J. Stubbe			4. DATE REPORT COMPLETED MONTH YEAR		
7. PERFORMING ORGANIZATION NAME AND MAILING ADDRESS (Include Zip Code) Tractione1 Rue de la Science 31 B-1040 Brussels, Belgium			6. DATE REPORT ISSUED MONTH YEAR October 1986		
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12. SUPPLEMENTARY NOTES			11a. TYPE OF REPORT b. PERIOD COVERED (Inclusive dates)		
13. ABSTRACT (200 words or less) The code assessment results in this report were based on all the available data (from plant computer and plant recordings) from the Doel-2 steam generator tube rupture incident of June 1979. The study revealed that the code is capable to simulate the observed phenomena reasonably well and can be used to predict similar transients. However, lack of precise boundary conditions does not allow one to quantify the error margin from this analysis.					
14. DOCUMENT ANALYSIS - a. KEYWORDS/DESCRIPTORS RELAP5/MOD2 steam generator tube rupture Doel-2 plant ICAP code assessment b. IDENTIFIERS/OPEN-ENDED TERMS				15. AVAILABILITY STATEMENT Unlimited	
				16. SECURITY CLASSIFICATION (This page) Unclassified (This report) Unclassified	
				17. NUMBER OF PAGES	
				18. PRICE	





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