

NUREG/IA-0132 CAMP00 1 International Agreement Report

Improvements to the RELAP5/MOD3 Reflood Model and Uncertainty Quantification of Reflood Peak Clad Temperature

Prepared **by** Bub Dong Chung, Young Jin Lee, Chan Eok Park, Sang Yong Lee, KAERI Young Seok Bang, Kwang Won Seul, Hho Jung Kim, **KINS**

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CAMP001 **International Agreement Report**

NUREG/IA-0132

Improvements to the RELAP5/MOD3 Reflood Model and Uncertainty Quantification of Reflood Peak Clad Temperature

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Abstract

Assessment of the original RELAP5IMOD3. 1 code against the **FLEGHT SEASET** series of experiments has identified some weaknesses of the reflood model, such as the lack of a quenching temperature model, the shortcoming of the Chen transition boiling model, and the incorrect prediction of droplet size and interfacial heat transfer. Also, high temperature spikes during the reflood calculation resulted in high steam flow oscillation and liquid carryover. An effort had been made to improve the code with respect to the above weakness, and the necessary model for the wall heat transfer package and the numerical scheme had been modified. Some important **FLECHT-SEASET** experiments were assessed using the improved version and standard version. The result from the improved RELAP5/MOD3.1 shows the weaknesses of RELAP5/MOD3.1 were much improved when compared to the standard MOD3.1 code. The prediction of void profile and cladding temperature agreed better with test data, especially for the gravity feed test. The scatter diagram of peak cladding temperatures (PCTs) is made from the comparison of all the calculated PCTs and the corresponding experimental values. The deviation between experimental and calculated PCTs were calculated for 2793 data points. The deviations are shown to be normally distributed, and used to quantify statistically the PCT uncertainty of the code. The upper limit of PCT uncertainty at **95%** confidence level is evaluated to be about 99K.

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Summary

In the past decade, the benefit of best-estimate methodology rather than artificial conservative approach for the **LOCA** analysis have become obvious to the industry and regulatory body. In August **1988,** the Nuclear Regulatory Commission (NRC) approved the final version of a revised rule on the acceptance of emergency core cooling systems **(ECCS).** The revised rule contains that an alternate **EGGS** performance analysis, based on best estimate methodology, may be used to provide more realistic estimates of plant safety margins. According to the preliminary studies, the new methodology of best-estimate analysis is expected to substantial **LOCA** margin gains over the traditional conservative analysis. Its overall benefit could be translated into reduced costs of million of dollars per year to utility.

This research aims to develop reliable, advanced system thermal-hydraulic computer code and to quantify the uncertainties of code to introduce the best-estimate methodology. One of the best estimate code, RELAP5, has been developed jointly **by** the NRC and a consortium consisting of several countries and organizations that are members of the International Code Assessment and Application Program (ICAP). The code is being continually updated and recently the RELAP5/MOD3.1 Version has been released after a beta-testing of RELAP5/MOD3 version **7j.** Although the emphasis of the RELAP5/MOD3 development was on large-break LOCAs, several deficiencies in its reflood model were identified during the independent assessments of the code as part of RELAP5/MOD3-KAERI Version development.

Some improvements to the RELAP5IMOD3 reflood model have been made. These improvements were made to correct deficiencies in the reflood model identified **by** the assessment of the R.ELAPS/MOD3 code against **FLECHT-SEASET** experiments. The improvements consist of modification of reflood wall heat transfer package and adjusting the droplet size in dispersed flow regime. The time smoothing of wall vaporization and level tracking of transition flow are also added to eliminate the pressure spikes and level oscillation during reflood process. Assessment of the improved model against **FLECHT-SEASET** experimental data and application of LBLOCA analysis for plant shows that the deficiencies have been corrected. The associated uncertainty is statistically quantified using the **FLECHT-SEASET** data. The selected test runs include a gravity feed test and several forced feed tests with wide range of the parameters such as flooding rate, system pressure,

initial clad temperature, rod bundle power. The results show that the code under-predicts the peak cladding temperature **by 7.56** K on average. The upper limit of the associated uncertainty at **95%** confidence level is evaluated to be about **99** K, including the bias due to the under-prediction.

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Nomenclatures

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1. Introduction

The postulated loss-of-coolant accident **(LOCA)** of a pressurized water reactor has been the subject of intensive experimental and analytical studies in light water reactor. Many efforts are devoted to the investigation of thermodynamic behavior of reactor core and effectiveness of emergency core cooling system during reflood phase of **LOCA.**

The series of RELAP code began with RELAPSE, which was released in **1966.** Subsequent versions of this code are RELAP2[1], RELAP3[2], and RELAP4[3]. **All** of these codes were based on the homogeneous equilibrium model (HEM) for describing the two-phase flow process. In **1976,** the development of a nonhomogeneous, nonequilibrium. model was undertaken for RELAP4. It soon became apparent that a total rewrite of the code was required to efficiently accomplish this goal. The result of this effort was the beginning of the RELAP5 project.

The principal new feature of the RELAP5 series [4,5] is the use of a two-fluid, nonequilibrium, nonhomogeneous, hydrodynamic model for transient simulation of the twophase system behavior. The MOD3 version of RELAP5 has been developed jointly **by** the NRC and a consortium consisting of several countries and organizations that are members of the International Code Assessment and Application Program (ICAP). The mission of the RELAP5IMOD3 development program was to develop a code version suitable for the analysis of all transients and postulated accidents in PWR systems, including both largeand small-break loss-of-coolant accidents (LOCAs) as well as the full range of operational transients. The code is being continually updated and recently the MOD3. 1 version of RELAP5 **[6]** has been developed jointly **by** the NRC and a consortium of International Code Assessment and Application Program (ICAP). Although the emphasis of the RELAP5/MOD3.1 development was on large-break LOCAs, several deficiencies in reflood model were identified during the assessment of **FLECHT-SEASET** series of experiments **(71.** The deficiencies are categorized as **1)** High pressure spikes and oscillation during reflood 2) Delayed quenching **3)** Incorrect void profile and vapor cooling in dispersed flow.

Parallel to the development of best-estimate code, the Nuclear Regulatory Commission (NRC) approved the final version of a revised rule on the acceptance of emergency core cooling systems **(ECCS)[8].** The revised rule contains that an alternate **ECCS** performance analysis, based on best estimate methodology, may be used to provide more realistic

estimates of plant safety margins. However the licensee must quantify the uncertainty of the estimates and includes that uncertainty when comparing the calculated results with acceptance limits. To support the revised **ECCS** rule, the NRC research formed a small group of experts, called the Technical Program Group(TPG). The TPG developed a method called the Code Scaling, Applicability, and Uncertainty **(CSAU)** evaluation methodology **[9- 15]** and demonstrated for Westinghouse four-loop pressurized water reactor with 17x17 fuel using TRAC-PF1/MOD1 code.

The purpose of this study is to present a reflood model and its implementation in RELAP5/MOD3. **1. A** great deal of effort has been made to solve to the above deficiencies, and the necessary model improvement and code modification has been carried out. The modified reflood model was assessed using **FLECHT-SEASET** test and it's uncertainty was also quantified.

2. Reflood Model Improvements

The model modification and development activity were focused on solving the RELAP5/MOD3.1 model deficiencies. Followings are suggested to be the primary cause for the fore-mentioned code deficiencies.

- a) Unsuitable **CHF** correlation for low pressure and low flow. Discontinuity in the wall heat transfer logic
- **b)** Lack of quenching temperature model
- **c)** Lack of droplet field model in dispersed flow

A great deal of effort has been made to improve the code with respect to the above causes, and the necessary model improvement and code modification has been carried out.

Unlike RELAP5/MOD2, RELAP5/MOD3.1 uses the same heat transfer coefficient logic for all wall surfaces. To avoid discontinuities, reflood surfaces are treated as regular surfaces, thus there is no reflood specific model. Structures flagged as reflood structures differ only in that axial conduction is considered. **A** boiling curve is used in RELAP5/MOD3.1 to govern the selection of heat transfer correlation's. In particular, the heat transfer regimes modeled are classified as pre-CIIF and post-CHF regimes. Condensation heat transfer is also modeled, and the effects of noncondensable gases are modeled. The heat transfer package in RELAP5/MOD3.1 uses heat transfer correlation's that are based on fully developed flow, where entrance length effects are not considered except for the calculation of **CHF.** The approach of using these correlation's in a transient code such as RELAP is often refereed to as the quasi-steady approach.

The following list gives the modes **by** which heat is transferred between heat structure surfaces and the fluid in contact with the heat structure.

mode 0 ; Convection to noncondensable-water mixture

mode 1; Single-phase liquid convection at critical and super critical pressure

- mode 2; Single-phase liquid convection at subcritical pressure
- mode 3 ; Subcooled nucleate boiling

mode 4 : Saturated nucleate boiling

mode 5 ; Subcooled transition film boiling

mode 6 ; Saturated transition film boiling

mode 7 ; Subcooled film boiling

mode 8 ; Saturated film boiling

mode 9 ; Single-phase vapor convection

mode **10;** Condensation when void equals one

mode **11;** Condensation when void is less than one

If the noncondensable quality is greater than **0.0001,** then 20 is added to the node number. If the heat structures are flagged as reflood structure, 40 is added thus the mode number can be 40 to *51.* Figure **1** is a schematic diagram showing the logic built into the code to select the appropriate heat transfer mode.

In the modified version, the above wall heat transfer packages were updated when reflood begins. Time smoothing of wall vaporization and level tracking of transition flow are also added to eliminate pressure spikes and level oscillation during reflood process. More detailed model descriptions are provided in the following section.

2.1 Wall Heat Transfer Package

The heat transfer package consists of a library of heat transfer correlation's and selection logic algorithm similar to RELAP5/MOD3.1. For the normal heat structures, the correlation and logic algorithms are exactly the same as those installed in RELAP5/MOD3.l. However when the heat structures are flagged as reflood structure, some modification of correlation's and logic algorithm are performed as shown in Figure 2. The modified correlation's used in each heat transfer regimes are detailed below.

2.1.1 **Critical Heat Flux and Transition Boiling**

In RELAP5, the transition boiling correlation is based on Chen transition boiling model **[116]** which is applicable to a dispersed flow regime. The model depends on the Critical Heat Flux **(CHF)** value and used to determine whether the film boiling occurs. Thus

GHF correlation is important in determining the flow regime. The Groeneveld Look up table **[17]** was used to determine the **CHF.** Unfortunately, the value in the table was found to change suddenly with respect to flow and quality at low pressure and low flow condition. It may result in numerical instabilities or oscillation. Modified wall heat transfer package is based on the heat transfer logic developed on the basis of wall temperature. The reflood heat transfer package is similar to RELAPS/MOD2 *[5]* and based' on the comparative study of post-GHF wall heat transfer package of RELAP5 codes which was done at Paul Scherrer Institute **(PSI),** Switzerland **[18].**

The intersection of the nucleate boiling and transition boiling heat transfer regimes occurs at the **CHF** point. To provide for a continuous transition between regimes, the **CHF** point **(q"CHF,** TCHF) must be specified. The modified Zuber pool boiling **CLIF** correlation *[5,* **f' 9]** is chosen as a reasonable approximation of the maximum heat flux at the quench front:

$$
q_{CHF} = \frac{\pi (1 - \alpha_{\kappa})}{24} h_{f_{\kappa}} \rho_{\kappa}^{0.5} [g \sigma (\rho_f - \rho_{\kappa})]^{0.25}
$$
 (1)

To define the boiling curve, it is necessary to know the surface temperature at which **CHF** occurs. An iterative procedure *[5]* is used to find the wall temperature at which the heat flux from Chen nucleate boiling correlation is equal to the critical heat **flux.** Thus,

$$
q_{CHEN}^{\dagger}(T_{CHF}) = q_{CHF}^{\dagger} \tag{2}
$$

The transition boiling regime is bounded **by** the **GHIF** point, (below which the wall is continuously wetted and nucleate boiling exits): and the minimum stable film boiling point (above which the liquid cannot wet the wall and film boiling exits). The minimum stable film boiling temperature is called sometimes rewetting or quenching temperature. There are several correlation's, i.e., Dix **&** Anderson [20] **,** Murao [21], Berenson [22] and Henry **[23]** correlation. Good agreement between several **FLECHT-SEASET** data [24] and predicted rewetting temperature was obtained when a formnulation of Henry correlation was used [25]. Thus Henry correlation is incorporated in modified RELAP version to determine the minimum stable film boiling temperature and has following form:

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$$
T_{MN} = T_{MN,B} + 0.42(T_{MN,B} - T_I) \{ \left[\frac{(k \rho C_p)_I}{(k \rho C_p)_w} \right]^{0.5} \frac{h_{fv}}{C_{pw}(T_{MN,B} - T_f)} \}^{0.6}
$$
 (3)

$$
T_{\text{MIN},B} = T_f + 0.127 \frac{\rho_r h_{f_k}}{k_r} \left[\frac{g(\rho_f - \rho_g)}{(\rho_f + \rho_g)} \right]^{2/3} \left[\frac{\sigma}{g(\rho_f - \rho_g)} \right]^{1/2} \left[\frac{\mu_r}{g(\rho_f - \rho_g)} \right]^{1/3}
$$

At present, there is no consensus on a correlation to use for the transition boiling regime. Modified version employs a simple interpolation scheme for heat transfer between **CHF** temperature and minimum film boiling temperature.

$$
q_{TRAN} = \dot{q}_{CHF} + (1 - \delta^2) \dot{q}_{FB}
$$

where δ is defined as $(T_w - T_{MN}) / (T_{CHF} - T_{MN})$ (4)

The above mentioned heat flux should be partitioned to the liquid and the vapor phase for two fluid model. Assuming that the heat transfer coefficient of vapor side does not change much, the energy partition of transition region can be estimated as follows.

$$
h_g = h_{g\text{CHF}} + (1 - \delta)h_{g,\text{FB}}
$$

\n
$$
q_g = h_g(T_w - T_g)
$$

\n
$$
q_f = q_{\text{TRAN}} - q_g
$$
\n(5)

2.1.2 Film Boiling

Film boiling is described **by** heat transfer mechanisms that occur during several flow patterns, namely inverted annular flow, slug flow and dispersed flow. The wall-to-fluid heat transfer mechanisms are conduction across a vapor film blanket next to a heated wall, convection to flowing vapor and between the vapor and droplets, and radiation across the film to a continuous liquid blanket or dispersed mixture of liquid droplets and vapor.

The single phase vapor correlation's become the model basis of the convection heat transfer in film boiling mode. However the presence of the droplet in steam flow provides a source of turbulence additional to that generated **by** wall shear, and this will enhance the steam convective heat transfer as deduced from steam-only experiments **[26].** Several investigators have looked at the effect of turbulence intensity on convective heat transfer in

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two-phase dispersed flows. Drucker et al. **[27]** proposed that the droplets will enhance turbulence in the flow **;** hence, heat transfer. The ratio of the two-phase-to-the-single-phase heat transfer coefficient ϕ can be written for entrained flow as

$$
\varphi = \frac{h_{TP}}{h_{SP}} = 1 + 3.25 \left\{ \frac{(1 - \alpha_g)Gr}{Re^2} \right\}^{0.5}
$$
 (6)

where $(1-\alpha_g)$ represents the liquid fraction and Grashof number, Gr, and flow Reynolds number, Re, based on steam properties and defined **by**

$$
Gr = \frac{g(\rho_f - \rho_g)\rho_g D_H^3}{\mu_g^2} \tag{7}
$$

and

$$
Re = \frac{\rho_g V_g D_H}{\mu_g} \tag{8}
$$

The above two phase enhancement effects are included in the convection term **(** Dittus Boelter Correlation) of the film boiling mode. Similar enhancement effects are included in other codes, COBRA-TF **[28]** and Westinghouse BART **[29].** The correlation's in other codes, COBRA-TF [28] and Westinghouse BART [29]. The correlation's in
RELAP5/MOD3 conduction (modified Bromley Correlation) and radiation model are deemed sufficiently accurate and are not changed.

2.2 Wall Vaporization Smoothing Model

In RELAP5/MOD3, there are two interphase mass transfer terms. One is a wall vaporization due to wall heat transfer and the other is a mass transfer arising from bulk exchange between the liquid and vapor spaces. The latter is treated as a partially implicit term, although the interfacial heat transfer coefficient is estimated explicitly. However the first term, wall vaporization, is treated as an explicit term in the mass and energy equation.

Fig. 1 Schematic of wall heat transfer logic

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Fig. 2 Schematic of reflood wall heat transfer logic

This scheme was found to cause numerical oscillation. It is well known that a numerical underrelaxation can prevent this kind of oscillations.

Thus time smoothing of wall vaporization is implemented to a modified version as **follows.**

$$
\Gamma_{w,n+1} = \eta \Gamma_{w,n} + (1 - \eta) \Gamma_{w,n+1}
$$
 (9)

The underrelaxation factor is of the form, $\eta = e^{-\Delta t/\tau}$, in order to obtain time-step insensitive smoothing. For reflood case $\tau = 0.1$ sec was selected because time constant for major transient phenomena is considered as longer than **0. 1** second.

2.3 Water Level Tracking Model for Transition Flow

Such codes as RELAP5 code which use Eulerian coordinate system for the solution of the finite difference equation, cannot track the two phase mixture level unless systems were modeled with very fine nodalization. Although a fine mesh nodalization of reflood heat structure is provided to account for the axial conduction, the lack of level tracking results in incorrect heat transfer coefficient for a fine mesh heat structure in a given coarse mesh hydro-cell. This impact is more severe for the developing flow.

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To circumvent this, a level tracking model is newly implemented in modified version for the calculation of the heat transfer coefficient of fine mesh heat structure. The variation of hydraulic parameters in a hydro-cell can be estimated with proper assumptions. One of the major parameters which govern the wall heat transfer is a void fraction. It is assumed that the void fraction in a hydro-cell has a step change between upper and lower void fraction of hydro-cell, while other parameters remain constant. The model is coded as a following equation.

$$
\alpha_{\kappa}(z) = \alpha_{\kappa} \qquad \text{if} \quad 0 < z < z_{level}
$$
\n
$$
\alpha_{\kappa} \qquad \text{if} \quad z_{level} < z < 1 \tag{10}
$$

, where α_K means the void fraction of downstream volume, α_L is void fraction of upstream

volume, and α_g is void fraction of given hydro-cell. The water level α_g is defined as $(\alpha_g - \alpha_L)/(\alpha_K - \alpha_L)$. The above scheme is activated when $\alpha_K < 0.1$ and $0.1 < \alpha_g < 0.9$, and only one of the cells related to a reflood structure is applicable.

2.4 Droplet Model for Dispersed Flow Regime

In RELAP5/MOD3, the bubbly and mist flow regimes are both considered as dispersed flow. The dispersed bubbles or droplets can be assumed to be spherical particles with a size distribution following the Nukiyama-Tanasawa form [30]. The average diameter d_0 is obtained by assuming that $dq = (1/2)d_{max}$. The maximum diameter, dmax, is related to the critical Weber number, We = $d_{\text{max}} \rho_c$ (vg-vf)²/ σ . The values for We are taken presently as **10.0** for bubbles and **3.0** for droplet. For reflood case, the value 12 was taken for droplet and average droplet size was restricted between 2.5 mm and hydraulic diameter (10 mmn for typical *PWR).*

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However estimated droplet size was too large comparing with the **FLEGHT-SEASET** experiment and COBRA-TF estimations **[31].** It results in too much liquid accumulation at the downstream of the quench front and incorrect vapor cooling, according to the PSI evaluation of reflooding model **[32].** In the modified version, there is no change in correlation's for interfacial drag and heat transfer, but the average droplet size for reflood case is restricted between 0.2 mm and 2.0 mm according to FLECHT experiment result. **All** interfacial surface area for mist flow regime were estimated based on the above droplet diameter. Similar restriction on droplet size for dispersed film boiling was proposed by PSI \cdot **[33]**

3. Model Verification

Runs **31504,31805,31302** and **31701** from the experiments of the 161-rod **FLEGHT-SEASET** facility were simulated to assess the reflood model of RELAP5/MOD3/KAERI at various reflood rates and also the run **33338** was simulated for the gravity driven reflood. The electrically heated rod configuration of **FLECHT** was typical of the full-length Westinghouse 17x17 rod bundle. The rod had a cosine axial power profile. This report includes the input deck model for **FLEGHT** forced feed and gravity feed reflood. The assessments of improved model for RELAP5/MOD3/KAERI were performed and the results were compared with the results obtained using the original RELAP5IMOD3. **1.** The overall performance of the code predictability was evaluated with respect to the peak cladding temperature and the quenching time.

The objective of **FLEGHT-SEASET** program is to provide experimental heat transfer and two phase flow data in simulated PWR geometry for postulated conditions of reflooding, core boil off, and natural circulation. **A** series of forced flow and gravity feed bundle reflooding: tests and steam cooling tests were conducted on a heater rod bundle whose dimensions are typical of the current PWR fuel rod array. The actual array configuration and dimensions of test heater rods are shown in Figure **3.**

The test parameters cover a spectrum of conditions that encompass both the best-estimate and current licensing calculations. These tests examined the effect of initial clad temperature, variable stepped flooding rates, rod peak power, constant low flooding rates, coolant subcooling, and system pressure. Table **1** shows the ranges of test parameters. Detailed descriptions of 161-rod **FLECHT-SEASET** are described in Reference **1. Of** the 161-rod tests, 4 forced feed reflood **(31302,** 31504, **31701, 31805)** and **1** gravity feed reflood **(33338)** cases, as shown in Table 2, were selected for the developmental assessment.

Fig. **3** Bundle Cross Section of **FLECHT-SEASET** Test Section

Test Run Number	Pressure (Mpa)	Maximum Clad Temperature (K)	Flooding Rate (cm/sec)	Injected Liquid Temperature (K)
31504	0.28	1136	2.4	324
31805	0.28	1144	2.1	324
31302	0.28	1142	7.65	325
31701	0.28	1145	15.5	326
33338	0.28	1144	Gravity	325

Table **1.** Assessment Matrix for **FLECHT SEASET 161** Rod Test

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Table 2 Test Matrix for Assessment

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3.1 Simulation Model

The test section was modeled using 20 uniform cells, as shown in Figure 4. Measured fluid conditions were used to define the thermal- hydraulic conditions in the upper and lower time-dependent volumes, which represented the upper and lower plena, respectively. The measured flow injection velocity was used to define the flow conditions at the timedependent junction that connects the lower plena and the test section. The measured power, which decreased during the test period, was used as input for heat structures representing the rods. If a modeled cell has a grid spacer, the head loss of spacer grid was considered **by** subtracting the spacer grid blockage from the normal junction. The spacer grid was also considered in the **CHF** calculation for heat structures. The heatup and reflood phases of tests were simulated in a single transient calculation using the measured heatup and decay power. The measured cladding temperatures before the heatup phase were used as the input for the initial temperatures required for each heat structures. The start time of water injection was also used as the input value.

The nodalization diagram for Gravity Feed Test is shown in Figure **5.** As shown in the Figure, the test section and heater rod model are the same as the forced feed simulation except the addition of downcomer and associated pipes. The downcomer was modeled as a pipe with **10** cells to predict the correct water level. The experimental reflood injection flow rate is applied to the time-dependent junction connected to the bottom of the downcomer and the connecting pipes and valves are also modeled. The measured flow rate injected to the bottom of downcomer was used to define the conditions at the time dependent junction connected to the bottom of downcomer.

Calculation is performed in the same way as in the forced reflood cases : one-through calculation of heat-up and reflood phase, initial condition using the experimental data at early heat-up phase. reflood feed trip at starting time of power decay, division of heater rod bundle according to the power distribution.

Assessments for base case and case for using modified version (RELAP5/MOD3/KAERI) ² were performed using same nodalization and same sequence of events. Also, there were no diviations; from the user guidelines in assessment. The one difference between-base and modified case is that the input deck of modified case have "Group **1** Options" which activate the modified model. See the Appendix **A** for the details of option used in modified version.

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3.2 Assessment Results

The experimental data for 161-rod **FLECHT SEASET** were obtained from the **ENCOUNTER** Data Bank of **USNRC** [34]. The raw experimental data contain information' s from the *256* channels of the data acquisition system. The data channels consist of **177** heater rod surface temperatures, system temperatures, bundle power, flows and absolute and differential pressures. However, some of the raw data from the measurement channels contained failed or spurious data which were rejected after inspection and in some cases **by** engineering decision.

The data for each test were sorted according to the measurement location and measurement type, and then used for the comparison of calculation results and uncertainty quantification of reflood PCT (Peak Clad Temperature).

As shown in Figure 4, the test section was nodalized into 20 equal size nodes using the 'PIPE' component. However, the axial measurement points are not spaced at regular intervals but concentrated in the mid-elevation region. This meant that for most axial measurement locations, a computational cell which accurately matches the given measurement location was not available. Thus an interpolation scheme for calculation results was necessary for the valid comparison between the calculation and the experimental data. In the present assessment, the assessment of forced feed and gravity feed test were performed based on a simple linear interpolation of the calculated results.

3.2.1 Forced Feed Test

In addition to the reference run (Test *31504),* three forced feed cases were simulated to investigate the capability of the code with respect to varying injection rates while keeping other parameters constant.

a) Test Run 31504 **-** Reference Test Run

On the reference test run *31504,* with an injection velocity of 2.46 cml/s **(0.97** in./s), the original RELAP5/MOD3. 1 code exhibited a number of weaknesses. In the comparison, all the experimental data at the same elevation excluding the failed channel data were averaged whereas the calculation results were linearly interpolated between the hydraulic cells that fence the measurement location.

Comparisons of averaged experiment data and calculated rod surface temperature histories are presented in Figure 6 (48 in. : low elevation) **.** 7 (72 in. : midplane), and 8 (96 in. : high elevation). The measured steam temperature at midplane (72 in.) is shown in Figure **9.** The predicted initial slope of the heatup rod temperature increase is accurate, but the calculated temperature turnaround occurs too early. As a consequence, the peak temperature at the midplane is underpredicted **by** about **50** K. This trend becomes more severe as the elevation increases. In the comparison of the steam temperature at midplane, the calculated steam temperature is much lower than the experimental data and this will contribute to the underprediction of PCT. The unsatisfactory prediction of the steam temperature may be caused **by** the inaccurate energy partition in the dispersed flow regime and the inaccurate interfacial heat transfer. In the modified version, an enhancement model of single phase heat transfer in dispersed flow regime is incorporated and this model contributed to the slight increase in the vapor temperature seen in Fig. **9.** However, the improvement in the PCT prediction is less than **10** K.

The calculated quenching behavior using the original version well illustrates the shortcomings of the reflood model. In the calculated results obtained with the original mod **3.1** code, there is an unrealistic 200 second quenching tail at the midplane. This is suspected to be caused **by** the Chen transition boiling model which yields values that are too small. In the modified version, a quenching temperature model (Modified Henry correlation) and a **Cl-F** temperature are used for determining the transition boiling heat transfer derived **by** interpolating between these two temperatures. These schemes resulted in great improvements in the prediction of the quenching behavior as shown in the Figures.

Comparison of inlet absolute pressure is presented in Figure **10.** Unrealistic large pressure spikes were calculated to occur with the original version at the time of quenching for each heat structure. The wall vaporization smoothing model and level tracking model for the developing flow were incorporated in the modified version which rectified these deficiencies and consequently the pressure trends were well predicted. These improvements were also found in exit steam and liquid flow as shown in Figures **11** and 12.

Figure **6.** Cladding Temperature at 48 in. elevation. Test 31504

Figure 7. Cladding Temperature at 72 in. clevation. Test 31504

Figure 8. Cladding Temperature at 96 in. elevation, Test 31504

Figure 9. Steam Temperature at 72 in. elevation, Test 31504

Figure 10. Pressure at 12 in. elevation, Test 31504

Figure 11. Exit Steam Flow, Test 31504

Figure 12. Exit Liquid Flow, Test 31504

Figure 13. Void Fraction at 67 in. elevation. Test 31504

The calculated void fractions near the midplane **(67** in.) is presented in Figure **13.** It shows that there is excessive liquid accumulation downstream of quenching front. This may be caused **by** the low predicted interfacial friction for the dispersed flow regime. Based on the **FLECHT** experimental observations, modified version the maximum diameter of droplet size was set at the value of 2.0 mm. This restriction contributed to increasing the interfacial drag in dispersed flow regime and improving the axial void profile. Figure 14 shows that the trend of collapsed liquid level of the test section is also much improved with the modified version.

b) Test Run **31701 -** High Injection Velocity

The calculated rod surface temperatures for run **31701** are presented in Figures **15** through **17** and the Figures show that the calculated results in the lower-to-middle elevation region agree well with the data. However, the results of the calculation with the original version for the upper elevation (Fig. **17)** show that the quenching of this section is calculated to occur too quickly. This is probably the result of high liquid fraction and high oscillatory steam velocity caused **by** the pressure spikes. Figure **16** shows this pressure spikes at the inlet of test section during high reflood injection. With the modified version, the pressure spikes are very much reduced and the void fraction and steam velocity are well predicted. As a result, the surface temperatures are predicted well with the modified version.

c) Test Run **31302 -** Medium Injection Velocity

In the medium reflood injection test, the main characteristics are similar to the high injection test (run **31701).** As shown in Figure **19** through 22, the rod surface temperature and hydraulic behaviors are well predicted with the modified version.

d) Test Run **31805 -** Very Low Injection Velocity

The main characteristics of the test run **31805** with a liquid injection velocity of 2.1 cml/sec are similar to reference test run (run 31504) which was conducted with the injection

Figure 14. Water Level. Test 31504

Figure **15.** Cladding Temperaturc at '18 in. clcvation. Test **31701 -**

Figure 16. Cladding Temperature at 72 in. elevation, Test 31701

Figure 17. Cladding Temperature at 96 in. elevation, Test 31701

Figure **18.** Inlet Pressure. Test **31701**

Figure **19** Cladding Temperature at 48 in. elevation, Test **31302**

Figure 21 Cladding Temperature at 96 in. elevation. Test 31302

Figure 23. Cladding Temperature at 48 in. elevation, Test 31805

velocity of 2.46 cm/sec. **All** thermal-hydraulic behaviors were delayed compared to the reference test due to the slightly lower injection velocity. The Figures **23 - 26** show that the quenching tail and pressure trends are improved with the modified version, although the rod surface temperature turnaround occurs too early due to the same reasons as in the reference test (31504) case.

3.2.2 Gravity Feed Test

Test run **33338** of gravity feed was selected for developmental assessment because it is a more realistic reflood situation. The radial power distribution was accounted for in the calculation and the rod surface temperature results from hot channel were presented in Figures **27** through **29.**

The surface temperature is reasonably predicted during the initial high reflood injection **(-15** second). After the reduction of reflood rate, the test data show a slight increase in temperature while the original code predicted the continuous decrease and early quenching. The deviations became greater in the middle-to-upper elevation. This weakness of original version is probably due to the incorrect void fraction and steam velocity in test section. Unlike the forced reflood case, the liquid flow entering in the test section depends on the small pressure difference between the downcomer and the test section. If pressure spikes are predicted to occur in calculation, these may greatly affect the liquid injection velocity. Figure **30** shows the pressure variation at the test section inlet and shows the severe pressure spikes calculated to occur with the original code. With the help of wall vaporization smoothing in the modified version, these pressure spikes were diminished after the deduction of reflood rate. This reduced the flow oscillation in test section (Figure **31)** and resulted in the correct prediction of rod surface temperature.

The quenching behavior of surface temperature were also predicted well in the modified version due to the quenching temperature model.

Figure 24. Cladding Temperature at 72 in. elevation, Test 31805

Figure 25. Cladding Temperature at 96 in. elevation, Test 31805

Figure 26. Pressure, Test 31805

Figure 27. Cladding Temperature at 48 in. elevation, Test 33338

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Figure 28. Cladding Temperature at 72 in. elevation, Test 33338

Figure 29. Cladding Temperature at 96 in. elevation, Test 33338

Figure 30. Inlet Pressure, Test 33338

Figure 31. Flow-rate between Downcomer and Core, Test 33338

3.3 Turn-around Temperature

In **FLECHT** experiment there are many radial measurement locations in same elevation. The test data are scattered due to many reasons; e.g., non-uniform manufactures of electric heaters, **2D/3D** effect of flow, and errors in the measuring calibration. To account for these measurement and hydraulic uncertainties the calculated turn-around temperature (i.e. peak clad temperature) at each measurement elevation was compared with all of the radial measurement channels available for that elevation. The scatter-graph of the calculated PCT of original version versus measured PCT is presented in Figure **32.** In this figure, the gravity reflood case was excluded in order to identify the effect of liquid injection rate on PCT. As shown in the Figure, the scattering band of test data is about **100** K. There is a general trend in PCT of a slight overprediction at low temperatures and an underprediction at high temperatures. It shows that the RELAP5/MOD3.1 underpredicts the clad temperature when the injection flow rate is low. With the modified version, although there is a slight improvement in low temperature region, nearly the same results were obtained. The Figure **33** shows the graph of calculation results versus test data the and uncertainty band.

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3.4 Quenching Time

The determination of quenching time depends on the definition of quenching. In this report the quenching time was defined as the latest time that the clad temperature reaches **500** K *(50* K above the **CHF** temperature). Such a simple definition enabled an easy comparison between the calculation result with the test data.

Figure 34 shows that the original code predicts early quenching in the case of high liquid injection and delayed quenching in low liquid injection case. The scattering of predicted results is too broad and it highlights the shortcomings in the transition boiling model and the problem of the flow oscillation due to pressure spikes during reflood. These weaknesses of the code were addressed and improved in the modified version and much better results were obtained as can be seen in Figure *35.*

Figure 32. Measured vs. Calculated PCT scatter-diagram for RELAP5/MOD3.1, Tests 31302, 31701, 31805 and 33338

Figure 33. Measured vs. Calculated PCT scatter-diagram for RELAPS/MOD3/KAERI, Tests 31302, 31701, 31805 and 33338

Figure 34. Measured vs. Calculated Quench-Time for
RELAP5/MOD3.1, Tests 31302, 31701, 31805 and 33338

Figure 35. Measured vs. Calculated Ouench-time scatter-diagram for
RELAP5/MOD3/KAERI, Tests 31302, 31701, 31805 and 33338

4. Model Assessment and Uncertainty Quantification

We focus our concern on assessing the reflood PCT predictability of RELAP5/MOD3/KAERI during LBLOCA and quantifying the associated uncertainty applicable to an LBLOCA realistic evaluation model(REM). For uncertainty quantification there should be a sufficiently largge number of available test data so that a statistical treatment may be possible, and the pool of data should cover the conditions expected to occur during, LBLOCA. **FLECHT SEASET** test is chosen because the test facility is **full** sized with respect to axial height and experiments were performed on wide ranges of test conditions. We compare the experimental and calculational PCTs for the forced and gravity feed reflooding of **161** rod unblocked bundle tests with variations of the parameters such as flooding rate, initial clad temperature, rod peak power, and system pressure. The code uncertainty evaluated from data comparison with the relevant experimental data could be an estimate of the uncertainty attributable to the combined effect of the reflood models and correlation's in the code, RELAP5/MOD3IKAERI.

4.1 Assessment

The experimental data for the assessment are selected from the 161-rod **FLEGHT SEASET** reflood test data in the data bank of **USNRC, ENCOUNTER[34].** The raw data consist of **177** heater rod surface temperatures, steam probe temperatures, rod bundle powers, flow rates, and absolute and differential pressures. The failed data in the total *256* channels are determined and rejected in the assessment. Linear interpolation of calculation results is necessary to correctly compare calculational and experimental data at the same elevation. (Fig. **36)**

The **18** selected test runs with wide range of several parameters including flooding rate, system pressure, initial clad temperature, rod bundle power, and others, are divided into *5* groups designed to investigate the effect of each parameters as shown in table **1.** and described below.

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RELAP5 Nodalization Measurement Location : Channel Number

Fig. **36** RELAP5 Nodalization versus Location of Measurement

Group 1 **-** effect of flooding, rate

Three planes of test section are chosen for the comparison of calculated and experimental cladding temperatures in the test run 31701: low plane(at 48 inch elevation), mid plane(at **72** inch elevation), and high plane(94 inch elevation). Fig. **37** shows the clad temperature comparisons at low, mid, and high planes. The experimental data at the same elevation except the failed channel data are averaged to be compared with the corresponding calculational value. The thicker line represents the averaged experimental cladding temperatures at the elevation, and the thinner line represents the corresponding calculational values. Calculation shows very good agreement with the experiment at mid plane, but tends to over-predicts the cladding temperatures at low and high planes. However it is noted here that the experimental data show a broad spread especially in the low and high region of test section. Fig. **38** compares the non-averaged experimental and calculational cladding temperatures at high plane. The line marked with solid square shows calculational value and the others represent experimental data. It can be seen that the predicted clad surface temperatures are within the scattered band of the experimental data. Therefore we can conclude that slight over-prediction of peak cladding temperature(PCT) shown from the curves for low and high planes in Fig. **37** is acceptable, and that RELAP5/MOD3IKAERI code well predicts the cladding temperature in the entire core for the high flooding rate experiment, **31701.** Fig. **39** shows the comparison of calculational and averaged experimental cladding temperatures at mid plane for various flooding rate. Calculation agrees well with the experiment in medium flooding rate(test runs, **31302** and **31203),** but slightly under- estimate the PCT in low flooding rate(test runs, 31504 and *31805).* And the under- prediction of PCTs in low flooding rate results from the early turn-around as shown in that figure.

Group 2 **-** effect. of system pressure

The test run, 31504 has been discussed in group **1,** but the results of the test runs, **32013** and 34209 are represented in Fig. 40. The calculational PCTs agree well with the experimental values both in the low and in the high pressures, but quenching is delayed in low pressure test, 34209. Then the delayed quenching may result in the over-prediction of

Fig. 37 Comparison of calculational and experimental cladding temperatures at selected elevations for test run, 31701

Fig. 38 Comparison of calculational and non-averaged experimental cladding temperatures at 96 inch elevation for test run, 31701

Fig. 39 Comparison of calculational and experimental cladding temperatures at 72 inch elevation for test runs, 31302, 31203, 31504, and 31805

Fig. 40 Comparison of calculational and experimental cladding temperatures at 72 inchelevation for test runs, 32013 and 34209

PCT in the down stream of core.

Group **3 -** effect of initial clad temperature

Experimental and calculational clad surface temperatures are compared in Fig. 41 for the test runs, 34420, **30817,** and **30518.** The test run **31203** was already discussed in group **1.** Somewhat early quenching appears in low initial clad temperature as shown in the curves for the test run, **30518,** but PCTs are little impacted **by** the early quenching. Therefore PCTs are well predicted even with the variation of initial clad temperature.

Group 4 - effect of rod bundle power

The test runs, **31021** and 34524 are presented in Fig. 42. The test run **31203,** which belongs to group **1** as well, is omitted here. The tests in the group 4 commonly show early turn-around behavior because of the low flooding rate. Quenching is delayed in high power as shown in the curves of test run, 34524, and it may result in over-prediction of PCT in the top region of core.

Group 5 - The other effect

The test run, **36026** is selected to analyze the effect of radial power distribution. It can be seen from the comparison of computational and experimental data in the radial high power region that the radial power distribution has no significant impact on PCT prediction. However, probably because the flooding rate is low, the code under-predicts the turnaround time. In the test runs, **32333** and **32235,** the flooding rate is varied during the transient. The RELAP5/MOD3JKAERI predicts well the PCTs even for the variable flooding rates. The delayed quenching in test run, **32235,** does not seem to be due to variable flooding rate, but due to the low system pressure. The test run, **31108,** is performed in low pressure and at medium flooding rate. The calculation shows good agreement with the experimental data. The low pressure does not delay the quenching in this test in contrast to the low flooding rate cases. The test run 34006 is characterized **by** low rod bundle power and low flooding rate. Early turn-around of clad surface temperature

 $Fig. 41$ Comparison of calculational and experimental cladding temperatures at 72 inch elevation for test runs, 30518, 30817, and 34420

Fig. 42 Comparison of calculational and experimental cladding temperatures at 72 inch elevation for test runs, 31021 and 34524

and early quenching appear in that test. These trends become more severe and result in large under-prediction of PCT in top region of core. The comparison plots for the test runs, 36026, 32333, 32235, 31108, and 34006, which were discussed above, are omitted here for brevity. The gravity feed test, which is closer to the reflood conditions, is conducted in test run, 33338. Radial power distribution is also allowed in this test. The predicted clad surface temperatures in the radially hot region are compared with the corresponding experimental values in Fig. 43. The behavior of cladding temperature including turn-around time, quenching time, and PCT, agree well with experiment in entire test section.

Fig. 43 Comparison of calculational and experimental cladding temperatures at selected elevations for the gravity feed test, 33338

4.2 Uncertainty quantification of Reflood PCT

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For the application to the best estimate methodology, the uncertainty associated reflood model should be quantified. The selected **FLEGHT** reflood test runs include a gravity feed test and several forced feed tests with wide range of the parameters such as flooding rate, system pressure, initial clad temperature, rod bundle power.

PCT is generally defined as the maximum of clad surface temperatures in the entire core region during the whole transient history. However this definition would require too many simulations in order to carry out statistically meaningful quantification of the uncertainty of PCT, because it produce only one value in a test run. We notice here that the clad surface temperature at PCT location is not the only important temperature, but those at other locations are also important to assess the code predictability of PCT. For practical purpose, PCT is then defined in this study as the local maximum value at a location of probe during the transient. Deviation between calculational and experimental PCTs is defined as follows.

$$
\Delta PCT_{Zi} = PCT_{Zi, \text{exp}} - PCT_{Z, \text{cut}} \tag{11}
$$

where subscripts, z means a elevation and subscript, zi means each measuring probe at the same elevation. In other word the highest temperature calculated **by** RELAP5 for a computational cell is paired with the highest temperature measured **by** a probe in that cell. Many thermo-couples share each computational cell, and the center of cell do not always coincide with the measurement location. Thus the linear interpolated calculational results at certain elevation are compared with the experimental data at that elevation. PCT bias is calculated **by** averaging all the available PGTs for assessment test matrix, and the upper limit of uncertainty of PCTs at *95* **%** confidence level is calculated **by** addition of the PCT bias to 1.645 times of standard deviation of all the $\triangle PCTs$, σ , under the assumption of normal distribution.

$$
\Delta PCT = Bias + 1.645 \sigma \tag{12}
$$

Figs. 44 to 48 show respectively the scatter diagram of PCTs for each test group. The xaxis represents PCT predicted **by** the code, and the y-axis does the experimental PCT. In these figures the solid line is the line of PCT bias, and the dashed line the upper limit of

PCT at 95% confidence level. For the group 1, which is constructed to investigate the effect of flooding, rate, the code under-predicts PCT **by** *18.65* K when compared with the averaged experimental PCT. The under-prediction of PCT is mainly due to the early turnaround in the tests with low flooding rate(test runs, 31504 and **31805).** The uncertainty of PCT for the group **1** is about **100** K, containing the **18.16** K bias. Since all the tests in group 2 have low flooding rates, they commonly show PCT under-prediction owing to early turn-around behavior. The run, 34209, which is a low pressure test, shows the delayed quenching and associated PCT over-prediction. As a result the bias in test group 2 is about the same as in test group **1,** but the uncertainty of PCT increases to about **130** K because of the combined effect of PCT under-prediction in low flooding rate and PCT overprediction in low system pressure. In the test group **3** the code predicts well the PCT for a wide range of initial clad temperature in spite of a little early or delayed quenching. The PCT bias is 3.64 K and uncertainty at *95* **%** confidence level is about 74 K, which are very low compared with the test group **1** and group 2. The code predicts well PCTs in test group 4 except the high power test run 34524. In this case the calculation shows delayed quenching and the consequent PCT over-prediction. The PCT bias of group 4 is -1.49 K and uncertainty of PCT is about **80** K. As discussed above, the radial power distribution, variable flooding rate, and gravity feed do not have significant effect on PCT predictability. Under-predicted in the low flooding rate tests, **36026** and 34006, and the over-predicted in the low pressure test, **32235,** and the well predicted PCTs in the other tests of group *5* are all combined and shown in Fig. 48. The PCT bias is **6.78** K and the corresponding uncertainty is about **108** K.

We collected the data from groups 1 to **5,** and constructed the scatter diagram of PCTs for all the test runs as shown in Fig. 49. The RELAP5/MOD3/KAERI code is shown to under-predict the PCTs **by** *7.56* K and the associated uncertainty including the bias are quantified to be **99.2** K. The validity of the assumption of normal distribution of PCTs is also checked **by** using the following ratio.

$$
R = |p - P| / \{p(1 - p)N\}^{1/2}
$$
\n(13)

where *P* and *P* respectively means the number of occurrence and its expected value from normal distribution, and **N** is the total number. The ratios are evaluated to be 0.12, **0.06,** and 0.16 in the outside of the bands, $(\mu + \sigma)$, $(\mu + 2\sigma)$, and $(\mu + 3\sigma)$, respectively, where μ denotes mean value and σ denotes standard deviation. Thus the assumption of normal distribution is valid because all the above ratios satisfy the general criteria, R **< 3 [35].**

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Fig. 44 Scatter diagram of calculational vs. experimental PCTs for test group **1:** effect of flooding rate

Fig. 45 Scatter diagram of calculational vs. experimental PCTs for test group 2: effect of system pressure

Fig. 46 Scatter diagram of calculational vs. experimental PCTs for test group 3: effect of initial clad temperature

Fig. 47 Scatter diagram of calculational vs. experimental PCTs for test group 4: effect of rod bundle power

Fig. 48 Scatter diagram of calculational vs. experimental PCTs for test group 5

Fig. 49 Scatter diagram of calculational vs. experimental PCTs for total test matrix

5. Run Statistics

All calculation against the **FLECHT SEASET** series of experiments had been performed using **HP-735** Workstation. For the reference run, 31504, the run statistics are summarized in Table **3.** The time step sizes and total **CPU** time is shown in Fig. **50** and **51** as a function of transient time.

		Standard RELAP5/MOD3 Modified RELAP5/MOD3
Total Simulation Time (sec)	900	900
Total CPU Time (sec)	4,247	5,030
Number of Time Steps	78,046	79,238
Number of Volumes	22	22
Grind Time (msec)	2.473	2.885

Table **3** Run Statistics for **FLECHT** Test Run 31504

In modified version, the grind time was increased **by 17%** comparing the original version because the modified version require more calculations in the implementation of new heat transfer logic, wall vaporization smoothing and level tracking model.

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Figure 50 Time Step Size and CPU Time Required for Original RELAP5

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Figure 51 Time Step Size and CPU Time Required for Modified RELAP5

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6. Conclusions

Assessment of original RELAP5/MOD3. 1 code against the **FLECET SEASET** series of experiments has identified some weakness of reflood model. The quenching of low reflood rate cases was delayed due to the lack of quenching temperature model and the shortcoming of Chen transition boiling model. Incorrect prediction of axial void profile and vapor cooling in dispersed flow resulted in increased cooling at the upper elevation. This was investigated to be caused **by** the incorrect prediction of droplet size and interfacial heat transfer. High pressure spikes during the reflood calculation resulted in the high steam flow oscillation and liquid carryover.

An effort had been made to improve the code with respect to the above weakness, and the necessary model for wall heat transfer package and numerical scheme had been modified. The weaknesses of *RELAP5IMOD3.1* were much improved in modified version. The prediction of void profile and cladding temperature agreed better with test data. These improvements are more dramatic for gravity feed test. In the application of plant LBLOCA analysis, it can be concluded that the predictability of modified version for whole thermal hydraulic behavior was reasonable and suitable for use as best estimate code for LBLOCA.

The scatter diagram of PCTs is made from the comparison of all the calculational PCTs and the corresponding experimental values. **2793** data in form of deviation between experimental and calculational PCTs are shown to be normally distributed, and used to quantify statistically the PCT uncertainty of code. The upper limit of PCT uncertainty at *95* **%** confidence level is evaluated to be about **99** K. The PCT uncertainty might be attributable to reflood models and correlation's in the code and experimental data spread. As mentioned above, the used data encompass so wide ranges of parameters that they cover the conditions expected to occur at reflood phase of LBLOCA. Therefore the evaluated uncertainty of reflood PCT could be applied to realistic evaluation model of LBLOCA.

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Appendix **A.**

Coding Change for RELAP5/MOD3/KAERI

 $\label{eq:2.1} \frac{1}{\sqrt{2}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2.$

 $\label{eq:2.1} \mathcal{L}(\mathcal{L}^{\text{max}}_{\mathcal{L}}(\mathcal{L}^{\text{max}}_{\mathcal{L}})) \leq \mathcal{L}(\mathcal{L}^{\text{max}}_{\mathcal{L}}(\mathcal{L}^{\text{max}}_{\mathcal{L}}))$ $\label{eq:2.1} \frac{1}{\sqrt{2}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2.$

 $\sim 3\%$

A. 1 Changes to Code Input

Reflood models described in Section **3** were implemented in the RELAP5/MOD3.1. In the modified version, the input in the user Group **1** input card is used to actuate a specific mode, and following are the added Group 1 card options.

GROUP 1 **CARD** OPTION

 $\label{eq:2.1} \mathcal{F}^{(1)}_{\mathcal{F}} = \mathcal{F}^{(1)}_{\mathcal{F}} = \mathcal{F}^{(1)}_{\mathcal{F}} = \mathcal{F}^{(1)}_{\mathcal{F}} = \mathcal{F}^{(1)}_{\mathcal{F}}$

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A.2. Subroutines Changed

The following subroutines are modified or added **by** the improvement. Listing of the each modified subroutine is available from the attached diskette. The whole content of the code modification is a part of KAERI's property and restricted to be used for the purpose of review at NRC and INEL only. The attached diskette also should be used only for the purpose of review on the code assessment report.

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 $\mathcal{L}^{\text{max}}_{\text{max}}$, where $\mathcal{L}^{\text{max}}_{\text{max}}$

 \mathcal{L}_{max} and \mathcal{L}_{max}

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A.3 New Variables Added

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The following are new variables added to the volume common block, voldat

a common block, vol2 is newly added due to droplet transport model. The variables in vo12 are as follow.

- system ordinal number i:
- $j:$ volume ordinal number

 \mathbf{i}

 $\label{eq:2.1} \frac{1}{\sqrt{2}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2.$

 $\label{eq:2.1} \frac{1}{\sqrt{2}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2.$ $\label{eq:2.1} \frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^{2} \left(\frac{1}{\sqrt{2}}\right)^{2} \left(\$

Appendix B.

Estimation of **FLECHT SEASET** Experimental Data Error

 $\label{eq:2.1} \frac{1}{\sqrt{2}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2.$

 $\frac{1}{2}$, $\frac{1}{2}$

 $\label{eq:2.1} \frac{1}{\sqrt{2}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2.$

 $\mathcal{L}^{\text{max}}_{\text{max}}$, $\mathcal{L}^{\text{max}}_{\text{max}}$

The instrumentation error associated with the data from **FLEGHT SEASET** unblocked bundle test series were derived either from equipment manufacturers' specifications or system calibration data. The data channel number and it's brief descriptions are shown in Table B.1. Table B.2 is a detailed listing of errors **by** data channel and run number. The standard deviation of best estimate of error is presented in Table B.2. The maximum possible error is also presented in Table. This is the sum of all possible component errors and is the outer bound of error. Detail explanation of the error analysis is presented in Appendix **D** of **FLEGHT** Data Report (Ref. **7).**

TABLE B.1

INITIAL **DATA ACQUISITION** SYSTEM **CHANNEL ASSIGNMENTS**

a. See paragraph **3-16**

b. See paragraph **3-17**

TABLE B.1 (cont) INITIAL DATA ACQUISITION SYSTEM CHANNEL ASSIGNMENTS

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TABLE B.1 (cont) INITIAL DATA ACQUISITION SYSTEM CHANNEL ASSIGNMENTS

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INITIAL **DATA** ACQUISITION SYSTEM **CHANNEL ASSIGNMENTS**

INITIAL **DATA ACQUISITION** SYSTEM **CHANNEL ASSIGNMENTS**

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 $\sim 10^{11}$

INITIAL DATA ACQUISITION SYSTEM CHANNEL ASSIGNMENTS

INITIAL DATA ACQUISITION SYSTEM CHANNEL ASSIGNMENTS

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INITIAL DATA ACQUISITION SYSTEM CHANNEL ASSIGNMENTS

INITIAL DATA ACQUISITION SYSTEM CHANNEL ASSIGNMENTS

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INITIAL **DATA** ACQUISITION SYSTEM **CHANNEL ASSIGNMENTS**

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TABLE B.2 INSTRUMENTATION ERRORS

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a. Refer to table 3-1 for identification of channels and functions.

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 $\mathcal{L}_{\mathrm{max}}$

b. All of these run numbers were applicable to these sensors, even though certain tests did not require certain transducers.

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$\frac{1}{2}$ TABLE B.2 (cont) INSTRUMENTATION ERRORS

 $B-14$

TABLE B.2 (cont) INSTRUMENTATION ERRORS

 $\frac{1}{\sqrt{2\pi}}\int_{0}^{\pi}d\mu\left(\frac{d\mu}{d\mu}\right) d\mu\left(\frac{d\mu}{d\mu}\right) d\mu.$

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 ~ 0.1

 $\sim 10^{-1}$

 ~ 100

Appendix C

RELAP5 Input Listings of Base Case and Modified Version for FLECHT-SEASET Test 31504

A single input deck except Group 1 Card (the card number is 1) was used to both the Standard RELAP5/MOD3.1 code and the KAERI-Modified code. *In the following RELAPS Input Listing, Group 1 Card was not used in standard RELAPS/MOD3. 1 calculation, while three words " 70, 73, 75" were added at Group 1 Card for modified code calculation*

 $\label{eq:2.1} \frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^{2} \left(\frac{1}{\sqrt{2}}\right)^{2} \left(\$

 $\label{eq:2.1} \frac{1}{\sqrt{2\pi}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2\pi}}\left(\frac{1}{\sqrt{2\pi}}\right)^2\frac{1}{\sqrt{2\pi}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{$

 $\mathcal{L}_{\mathcal{A}}$

 $\label{eq:2.1} \frac{1}{\sqrt{2}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2.$

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20800050 dt 0 *** contrl variables ** core collapsed water level 20520000 wt-level sum 1.0 0.0 1 20520001 0.0 0.18288 voidf 200010000 20520002 0.18288 voidf 200020000 20520003 0.18288 voidf 200030000 20520004 0.18288 voidf 200040000 20520005 0.18288 voidf 200050000 20520006 0.18288 voidf 200060000 20520007 0.18288 voidf 200070000 20520008 0.18288 voidf 200080000 20520009 0.18288 voidf 200090000 20520010 0.18288 voidf 200100000 20520011 0.18288 voidf 200110000 20520012 0.18288 voidf 200120000 20520013 0.18288 voidf 200130000 20520014 0.18288 voidf 200140000 20520015 0.18288 voidf 200150000 20520016 0.18288 voidf 200160000 20520017 0.18288 voidf 200170000 20520018 0.18288 voidf 200180000 20520019 0.18288 voidf 200190000 20520020 0.18288 voidf 200200000 ** water carry-over 20523000 flowrate mult 0.0116574 0.0 1 20523001 voidfj 250000000 20523002 rhofi 250000000 20523003 velfi 250000000 20521000 w-inte integral 1.0 0.0 1

 $\mathbf{u}_1 =$

20521001 cntrlvar 230 \bullet ** steam flow rate 20522500 stflow mult 0.0116574 0.0 1 20522501 voidgi 250000000 20522502 rhogi 250000000 20522503 velgi 250000000 20522000 s-inte integral 1.0 0.0 1 20522001 cntrlvar 225 $\star\star$ ** total power 20525000 power function 1.0 0.0 1 20525001 time 0 200 à. \bullet \star **TRIP LOGICS** ÷ 500 time 0 gt null 0 71.0 1 -1.0 501 httemp 200101008 gc null 0 1129.0 1 -1.0 $\ddot{\bullet}$ $***$ lower plenum 1000000 lplenum tmdpvol 1000200 3 500 1000201 0.0 2.8e5 324.15 1000202 1000.0 2.8e5 324.15 \star

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 \star

2000407 2.67366e-3 *9* *** *inlet* junction 2000408 **2.84762e-3 11** 2000409 **2.67366e-3** 12 *1500000* inlet tmdpJun 2000410 **2.84762e-3** 14 *1500101* **100000000** 200000000 **0.0116574** 2000411 2.67366e-3 *15* **1500200 1** *500* 2000412 2.84762e-3 **17 1500201 -1.0 0.0 0.0 0.0** 2000413 **2.67366e-3 18 1500202 0.0 0.378318 0.0 0.0** 2000414 2.84762e-3 **19 1500203 1000.0 0.378318 0.0 0.0** 2000415 2.67366e-3 20 **2000601 90.0** 20 *vol. vertical orientation **2000701 0.18288** 20 ***** vol. elevation change ** core **2000801** 1.e-6 **0.009731** 20 ***** vol. friction data **2000901** 1.14 1.14 1 ***** jun. loss coefficient **2000902 0.0 0.0** $\overline{2}$ $\overline{\mathbf{3}}$ 2000000 core pipe **2000903** 1.14 1.14 2000001 20 2000904 **0.0 0.0** 5 2000101 **0.0** 20 *vol. flow area **2000905** 1.14 1.14 6 2000201 **0.015571** 2 ***** jun. flow area **2000906 0.0 0.0 1** 2000202 **0.0116574 3- 2000907** 1.14 1.14 9 **2000203-0.015571 5** $\Delta \sim 10^7$ **2000908 0.0 0.0 1** 2000204 **0.0116574** *6* **2000909** 1.14 1.14 12**2000205 0.015571 8 1** 4 **2000910 0.0 0.0 2000206 0.0116574 9 2000911** 1.14 1.14 **151 7 2000912 0.0 0.0 2000207 0.015571 11 2000208 0.0116574** 12 **2000913** 1.14 1.14 **181 9 2000209 0.015571** *14* 2000914 **0.0 0.0** 2001001 **00100** 20 ***** vol. control flag 2000210 **0.0116574 15** 2001101 **000000 19** 2000211 **0.015571 17 *** jun. control flag 2001201 2.8e5 410.0 **0.0 0.0 0.0 333333333** 2000212 0.0116574 **18** 2001202 2.8e5 410.0 **0.0 0.0 0.0 2000213 0.015571 19** *vol. length **2001203** 2.8e5 410.0 **0.0 0.0 0.0 32000301 0.18288** 20 2000401 **2.67366e-3 1** 2001204 2.8e5 410.0 **0.0 0.0 0.0** 4*vol. volume 2000402 2.84762e-3 2 **2001205** 2.8e5 410.0 **0.0 0.0 0.0 *** vol. volume **-** grid volume2000403 2.67366e-3 **3 2001206** 2.8e5 410.0 **0.0 0.0 0.0 6**2000404 2.84762e-3 **5 2001207** 2.8e5 410.0 **0.0 0.0 0.0 7**2000405 2.67366e-3 *6* **2001208 2.8e5** 410.0 **0.0 0.0 0.0 2001208 3 2.8e5 410.0 0.0 0.0 0.0 8
2001209 3 2.8e5 410.0 0.0 0.0 0.0 9.0** 2000406 2.84762e-3 **8**

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2001210 3 2.8e5
410.0 0.0 0.0 0.0 10
2001211 3 2.8e5
410.0 0.0 0.0 0.0 11
2001212 3 2.8e5
410.0 0.0 0.0 0.0 12
2001213 3 2.8e5
410.0 0.0 0.0 0.0 13
2001214 3 2.8e5
410.0 0.0 0.0 0.0 14
2001215 3 2.8e5
410.0 0.0 0.0 0.0 15
2001216 3 2.8e5
410.0 0.0 0.0 0.0 16
2001217 3 2.8eS
410.0 0.0 0.0 0.0 17
2001218 3 2.8e5
410.0 0.0 0.0 0.0 18
2001219 3 2.8e5
410.0 0.0 0.0 0.0 19
2001220 3 2.8e5
410.0 0.0 0.0 0.0 20
2001300 0
2001301 0.0 0.0 0.0 19 * jun. initial condition
*2001401 0.0027353 0.0 1.0 1.0 1 *junc. hyd. diameter
*2001402 0.009731 0.0 1.0 1.0 2
*2001403 0.0027353 0.0 1.0 1.0 3
*2001404 0.009731 0.0 1.0 1.0 5
*2001405 0.0027353 0.0 1.0 1.0 6
*2001406 0.009731 0.0 1.0 1.0 8
*2001407 0.0027353 0.0 1.0 1.0 9
*2001408 0.009731 0.0 1.0 1.0 11
*2001409 0.0027353 0.0 1.0 1.0 12
*2001410 0.009731 0.0 1.0 1.0 14
*2001411 0.0027353 0.0 1.0 1.0 15
*2001412 0.009731 0.0 1.0 1.0 17
*2001413 0.0027353 0.0 1.0 1.0 18
*2001414 0.009731 0.0 1.0 1.0 19
2001401 0.009731 0.0 1.0 1.0 19
** outlet junction
2500000 outlet sngljun
2500101 200010000 300000000 0.0116574 1.34 1.34 101000
2500102 1.0 1.0 1.0
```
***25001 10 0.0027353 0.0 1.0 1.0**

```
2500201 0 0.0 0.0 0.0
\star** upper plenum
3000000 uplenum tmdpvol
3000101 1.0 1.0 0.0 0.0 0.0 0.0 0.0 0.0 00010
3000200 2
3000201 0.0 2.8e5 1.0
\star*** heat structure input***fuel rod
\bullet12001000 20 8 2 0 0.0 1 1 32
12001100 0 1
12001101 2 0.00122
12001102 1 0.00222
12001103 2 0.00411
12001104 2 0.00475
12001201 1 2 * boron nitride
12001202 2 3 * kanthal
12001203 1 5 * boron nitride
12001204 4 7 * ss 347
12001301 0.0 *2
12001302 1.0 3
12001303 0.0 7
12001400 -1
12001401 337.5 337.5 337.5 337.5 337.5 337.5 337.5 337.5
12001402 366.3 366.3 366.3 366.3 366.3 366.3 366.3 366.3
12001403 386.9 386.9 386.9 386.9 386.9 386.9 386.9 386.9
12001404 419.8 419.8 419.8 419.8 419.8 419.8 419.8 419.8
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 436.3 436.3 436.3 436.3 436.3 436.3 436.3 436.3 465.1 465.1 465.1 465.1 *465.1* 465.1 465.1 465.1 481.6 481.6 481.6 481.6 481.6 481.6 481.6 481.6 498.0 498.0 498.0 498.0 498.0 498.0 498.0 498.0 **502.1 502.1 502.1 502.1 502.1 502.1 502.1 502.1** 510.4 510.4 510.4 510.4 510.4 *510.4* 510.4 510.4 510.4 510.4 *510.4 510.4 510.4* 510.4 *510.4* 510.4 510.4 510.4 *510.4* 510.4 510.4 510.4 510.4 *510.4* **502.1 502.1 502.1 502.1 502.1 502.1** 502.1 **502.1** 493.9 493.9 493.9 493.9 493.9 493.9 493.9 493.9 12001415 477.4 477.4 477.4 477.4 477.4 477.4 477.4 477.4 *461.0* 461.0 461.0 461.0 *461.0* 461.0 461.0 *461.0* 451.1 451.1 451.1 451.1 451.1 451.1 451.1 451.1 12001418 440.4 440.4 440.4 440.4 440.4 440.4 440.4 440.4 436.3 436.3 436.3 436.3 436.3 436.3 436.3 436.3 428.1 428.1 428.1 428.1 428.1 428.1 428.1 428.1 20001000 **10000 1 1 29.07792** 20 200 0.02145 **0.0 0.0 3** 200 **0.03395 0.0 0.0** 4 200 0.04395 **0.0 0.0** *5* 200 *0.05545* **0.0 0.0 6** 200 0.06495 **0.0 0.0 7** 200 0.07445 **0.0 0.0 8** 200 **0.07995 0.0 0.0 9** 200 **0.08295 0.0 0.0 11** 200 **0.07995 0.0 0.0** 12 200 0.07445 **0.0 0.0 13** 200 0.06495 **0.0 0.0** 14 200 *0.05545* **0.0 0.0** *15* 200 0.04395 **0.0 0.0 16** 200 **0.03395 0.0 0.0 17** 200 0.02145 **0.0 0.0** 20 0.01 12 20.0 20.0 **0.0 0.0 0.0 0.0 1.0** 20 0.0112 0.09144 **3.56616** 0.09144 0.44196 1.14 1.14 0.43 0.0112 0.27432 **3.38328** 0.27432 **0.25908** 1.14 1.14 0.43 2 0.0112 0.4572 3.2004 0.45720 **0.07620** 1.14 1.14 0.43 **3**

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 0.01 12 0.64008 **3.01752 0.10668** 0.40132 1.14 1.14 **0.68** 4 0.0 112 **0.82296** 2.83464 **0.28956** 0.21844 1.14 1.14 **0.88 5** 0.01 12 1.00584 **2.65176** 0.47244 **0.03556** 1.14 1.14 **1.11 6** 0.01 12 **1.18872 2.46888** 0.14732 **0.38608** 1.14 1.14 **1.30 7** 0.01 12 **1.3716 2.286 0.33020 0.20320** 1.14 1.14 1.49 **8** 0.01 12 1.55448 **2.10312 0.51308 0.02032** 1.14 1.14 **1.60 9** 0.01 12 **1.73736** 1.92024 **0.16256 0.37084** 1.14 1.14 **1.66 10** 0.01 12 1.92024 **1.73734** 0.34544 **0.18796** 1.14 1.14 *1.66* **11** 0.01 12 **2. 103 12** 1.55448 **0.52832 0.00508** 1.14 1.14 **1.60** 12 0.01 12 **2.286 1.3716 0.17780 0.33020** 1.14 1.14 1.49 **13** 0.0 112 **2.46888 1.18872 0.36068** 0.14732 1.14 1.14 **1.30** 14 0.01 12 **2.65 176** 1.00584 **0.03556** 0.49784 1.14 1.14 **1.11** *15* 0.01 12 2. 834 64 **0.82296** 0.21844 0.31496 1.14 1.14 **0.88 16** 0.01 12 **3.0 1752** 0.64008 0.40132 **0.13208** 1.14 1.14 **0.68 17** 0.01 12 3.2004 0.4572 **0.05080** 0.45720 1.14 1.14 0.43 **18** 0.0 112 **3.38328** 0.27432 **0.23368** 0.27432 1.14 1.14 0.43 **19** 0.0112 **3.56616** 0.09144 **0.41656** 0.09144 1.14 1.14 0.43 20 **12001501 0 0 0 1 29.07792** ²⁰ $***$ $***$ housing 20 **3** 2 **0 0.097 0 0** 12002 100 **0 1** 2 **0.10208** 2 *ss304

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 337.5 337.5 337.5 366.3 366.3 366.3 386.9 386.9 386.9 419.8 419.8 419.8 436.3 436.3 436.3 465.1 *465.1* 465.1 481.6 481.6 481.6 498.0 498.0 498.0 **502.1 502.1 502.1** 510.4 510.4 510.4 510.4 510.4 510.4

12002413 **502.1** *502.1* **502.1** 12002414 *493.9 493.9* 493.9 1200*211* 477.4 477.4 477.4 12002713 *461.0* 461.0 461.0 12002417 451.1 *451.1* 451.1 12002418 440.4 440.4 440.4 12002410 11011 11011 1102 12002412 420.1 420.1 420. **12002501** 200010000 **10000 1 1 0.18288** 20 **12002601 0 0 0 1 0.18288** 20 **0 0.0 0.0 0.0** 20 **12002701 12002801 0.0** 20.0 20.0 **0.0 0.0 0.0 0.0 1.0** 20 **12002901 0.0** 20.0 20.0 **0.0 0.0 0.0 0.0 1.0** 20 $\star\star$ thimbles **¹²⁰⁰³⁰⁰⁰**20 **3** 2 **0** 0.005461 **0 1 12003000 20 4 ¹²⁰⁰³¹⁰¹**2 **0.0060198 12003101 2 0.0060198**
12003201 3 2 * ss 304 **12003201 0.0 C** 12003400 **-1** 12003401 **337.5 337.5 337.5** 1200310
2200210 **366.3 366.3 366.3 386.9 386.9** 1200*31*0 **386.9** 12003403-0002-0002 419.8 436.3 OIXIP 12003405 436.3 *465.1 465.1* 12003406 465.1 100.6 1200310
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منصد منصو 12003412 510.4 *510.4* 510.4 12003413 **502.1 502.1 502.1** 12003414 JUN
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12002412 510.4 *510.4* 510.4

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12004419 436.3 436.3 436.3
12004420 428.1 428.1 428.1
12004501 0 0 0 1 1.46304 20
12004601 200010000 10000 1 1 1.46304 20
12004701 0 0.0 0.0 0.0 20
12004801 0.0 20.0 20.0 0.0 0.0 0.0 0.0 1.0 20
12004901 0.0 20.0 20.0 0.0 0.0 0.0 0.0 1.0 20
\star\star12005000208 2 00.0 0 1
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12005100 0 1
12005102 1 0.00222
12005103 2 0.00411
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12005404 419.0 419.0 419.0 419.0 419.0 416.0 416.0 416.0 436.3
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12005400 403.1 403.1 403.1 403.1 404.6 406.00011 405.1 405.1 405.1 405.1 405.1 405.1 405.1 405.1 405.1 405.1 40
12005407 481.0 481.0 401.0 401.0 401.0 401.0 401.0 401.0 401.0 401.0 401.0 401.0 401.0 401.0 401.0 401.0 401.0
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12005413 **502.1 502.1 502.1 502.1 502.1 502.1** 502.1 **502.1** 12005414 493.9 493.9 493.9 493.9 493.9 493.9 493.9 493.9 12005415 477.4 477.4 477.4 477.4 477.4 477.4 477.4 477.4 12005416 461.0 461.0 461.0 *461.0* 461.0 *461.0* 461.0 461.0 12005417 451.1 451.1 451.1 451.1 451.1 451.1 451.1 451. 12005418 440.4 440.4 440.4 440.4 440.4 440.4 440.4 440. 12005419 436.3 436.3 436.3 436.3 436.3 436.3 436.3 436.3 12005420 428.1 428.1 428.1 428.1 428.1 428.1 428.1 428.1 **12005501 00 0 1 0.36576** 20 **12005601** 200010000 **10000 1 1 0.36576** 20 **12005701 0 0.0 0.0 0.0** 20 **12005801 0.0** 20.0 20.0 **0.0 0.0 0.0 0.0 1.0** 20 **12005901 0.0** 20.0 20.0 **0.0 0.0 0.0 0.0 1.0** 20 $***$ heat structure thermal property data 20100100 tbllfcttn 1 1 ***** boron nitride 20100200 tbilfctn **I I** *kanthal **20100300** tbl/fctn **I I** ss 304 20100400 tbl/fctn 2 2 *ss 347 $***$ thermal conductivity data 20100101 255.4 **25.584 533.2** 24.805 *boron nitride **337.5** 20100102 **810.9** 24.044 **922.0 23.732 20100103 1033.2. 24.420 1144.3. 23.12** 20100104 1255.4 **22.815 1366.5 22.503 20100105 1477.6 22.191 1588.7 21. 880** 20100201 255.4 **23.202 366.5 23.381** *kanthal **+** boron 20100202 **588.7 23.737 810.9** 24.094 **20100203 1033.2** 24 .450 1255.4 **24.807** 20100204 **1477.6** 25.164 **1588.7** 25.342 **20100301 255.4 12.060 266.5 15.931 *ss 30 20100302 922.0 23.092 1588.7 32.093** 20100401 255.4 **1600.** 13.064 0.0143 **0. 0. 0. 0. 273. 15** *ss 347 \ast

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volumetric heat capacity data
***20100151 255.4 1241374.1 533.2 2619098.6 * boron nitride
20100152 810.9 3315907.6 922.0 3486554.7
20100153 1033.2 3616213.0 1144.3 3714920.7
20100154 1255.4 3791042.7 1366.5 3847925.1
20100155 1477.6 3891423.3 1588.7 3924883.6
20100251 255.4 1880143.2 366.5 2368270.9
                                           * kanthal + boron
20100252 588.7 3209723.7 810.9 3777314.1
20100253 1033.2 4902562.0 1255.4 4224291.5
20100254 1477.6 4336390.6 1588.7 4384635.8
20100351 255.4 3593152.8 366.5 3828218.9
                                           * ss 304
20100352 922.0 4768483.2 1588.7 5843071.0
20100451 255.4 1600. 3541405.7 1668.0 0. 0. 0. 0. 273.15
                                                       * ss 347
\pmb{\ast}\star\starpower table
\bullet20220000 power 500 1.0 804578.3
20220001 -1.0 1.0
20220002 0. 1.0
20220003 1. 0.9962
20220004 2.5 0.9884
20220005 5. 0.9752
20220006 10.
               0.9493
20220007 15.
               0.9306
20220008 20.
               0.9110
20220009 25.
               0.8963
20220010 30.
               0.8817
20220011 40.
               0.8590
20220012 50. 0.8376
20220013 60. 0.8201
20220014 75. 0.7860
20220015 100.
               0.7484
20220016 125.
               0.7383
20220017 150.
               0.7040
20220018 175. 0.6835
20220019 200. 0.6665
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 $C-8$

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 $\mathcal{L}^{\text{max}}_{\text{max}}$, where $\mathcal{L}^{\text{max}}_{\text{max}}$

 $\frac{1}{2}$

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Federal Recycling Program

UNITED STATES NUCLEAR REGULATORY **COMMISSION WASHINGTON, DC 20555-0001**

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OFFICIAL BUSINESS PENALTY FOR PRIVATE **USE,** \$30 **SPECIAL STANDARD** MAIL **POSTAGE AND FEES PAID USNRC** PERMIT **NO. G-67**

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