ENCLOSURE 2

TENNESSEE VALLEY AUTHORITY BROWNS FERRY NUCLEAR PLANT (BFN) UNITS 1, 2, AND 3

TECHNICAL SPECIFICATIONS (TS) CHANGES TS-431 AND TS-418 -EXTENDED POWER UPRATE (EPU) - REPLACEMENT DOCUMENTATION (TAC NOS. MC3812, MC3743, AND MC3744)

CALCULATION MDQ099920060011, REVISION 1, "TRANSIENT NPSH/CONTAINMENT PRESSURE EVALUATION OF RHR AND CORE SPRAY PUMPS"

This enclosure provides TVA Calculation MDQ099920060011, Rev. 1, "Transient NPSH/Containment Pressure Evaluation of RHR and Core Spray Pumps."

The revised calculation utilizes the revised Appendix R results from TVA Calculation MDQ0999970046, Rev. 10 (Enclosure 1). Additionally, this calculation revision corrects an anomaly which existed at the two hour time point in the Appendix R wetwell pressure response and provides improved modeling of the values for required NPSH as a function of time (step function changed to continuous function) for the Appendix R event. The revised Appendix R results are shown in Figures 7.14 and 7.15.

TVAN CALCULATION COVERSHEET/CCRIS UPDATE

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The purp	ose	of this	calcula	ation is to	o deten	nine the	Net F	Positiv	e Suction	n Head (NF	PSH) av	ailab	le to the (Core Spray (CS) and
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pumps to	den	nonstr	ate that	adequa	ite mar	gins exis	st to e	nsure	that the	RHR and C	CS pum	ps pe	erform the	oir intended design
														. This calculation
(RAI) in s														TS-418 and TS-
431).														
	The results presented in Table 6.2-1 on page 21 show that adequate NPSH margins exist for each event scenario													
	analyzed. The minimum margins, maximum required containment (wetwell) overpressure, and the duration for required overpressure credit are presented in this table.													
Accortate														
	Acceptable results for the Appendix R event are based on assumed operator action at 2 hours to isolate all drywell coolers (see UNVERIFIED ASSUMPTION, Appendix R Assumption 6 on page 18).													
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TVAN CALCULATION COVERSHEET/CCRIS UPDATE

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CALC ID	TYPE	ORG	PLANT	BRANCH	NUMBE	R	REV	<u>ן</u>		
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KEY NOUNS (A-add, D-delete)

ACTION	KEY NOUN	<u>A/D</u>	KEY NOUN
(A/D)			
A	PUMP	Α	RHR
A	POOL	Α	CS
A	ATWS	Α	NPSH
A	DBA	Α	SBO
A	LOCA	Α	APPENDIX R

CROSS-REFERENCES (A-add, C-change, D-delete)

ACTION (A/C/D)	XREF CODE	XREF <u>TYPE</u>	XREF <u>PLANT</u>	XREF BRANCH	XREF <u>NUMBER</u>	XREF REV
Α	Р	CN	BFN	MEB	MDQ0999970046	R10
A	Р	CN	BFN	MEB	MDQ0023980143	R2
A	Р	CN	BFN	MEB	MDQ0064920353	R1
A	Р	VD	BFN	MEB	VTD-P160-0030	R6
A	S	CN	BFN	NTB	NDQ0999920116	R20
A	Р	VD	BFN	MEB	GE-ER1-AEP-06-334 (W79-060803-001)	
Α	Р	VD	BFN	MEB	VPF2647-10-1	
A	Р	VD	BFN	NTB	C1320503-6924	R2
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VA 40532 [07-20				Page 2 of 2		2-1 [07-08-2005]

	TION IDENTIFIER MDQ099920060011	
Title	TRANSIENT NPSH/CONTAINMENT PRESS OF RHR AND CORE SPRAY F	
Revision No.	DESCRIPTION OF REVISION	
0	Initial Issue	
	Total Number of Pages = 54 (including attachments)	
	The SAR and ISFSI SAR have been reviewed by <u>J.D. Wo</u> and this revision of the calculation does not affect SAR sect and 14.6 and does not affect any ISFSI SAR sections.	<u>lcott</u> tions <u>6.1, 6.2, 6.3, 6.4, 6.5, 14.5</u>
	Tech Specs and ISFSI CoC have been reviewed and deterr	nined not to be affected.
	The calculation reflects parameters/values associated with t Power Uprate (EPU) as well as the use of containment over calculating NPSH margins.	
1	This revision is a complete replacement for the previous rev additional Appendix R case with RHR pump flow rate of 900 of predecessor calculation MDQ0999970046. This revision Appendix R containment response data by GE which correct pressure transient at 2 hours due to an error in the initial cor- heat sinks at the start of the second phase of the analysis with	00 gpm consistent with Revision also incorporates revised ts a minor error in the wetwell indition specified for the wetwell
	Total Number of Pages = 55 (including attachments)	
	The SAR and ISFSI SAR have been reviewed by <u>A</u> and this revision of the calculation does not affect SAR sections.	MA ions <u>6.1, 6.2, 6.3, 6.4, 6.5, 14.5</u>
	Tech Specs and ISFSI CoC have been reviewed and determ	nined not to be affected.
	The calculation reflects parameters/values associated with the Power Uprate (EPU) as well as the use of containment over- calculating NPSH margins.	
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	TVAN CALCULATION TA	BLE OF CONTEN	ITS	
Calculation Ident	ifier: MDQ099920060011	Revision:	1	
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В	Recirculation Pump Motor Heat Load	1		(4 pages
С	Excel Spreadsheet Calculations		•••••••••••••••••••••••••••••••	(3 pages
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1	Sulzer Pumps Required NPSH Charts	•••••		(3 pages
2	Drywell Cooler Data			(7 pages
			Total Pages	55

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TVAN CALCULATION VERIFICATION FORM

Cal	culation Identifier MDQ09992	0060011			Revision 1	
Met	hod of verification used:					
1.	Design Review	\boxtimes				
2 .	Alternate Calculation		Verifier	Julie Jarvis/Bechtel	Date	
3.	Qualification Test					
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Comments:

This calculation was verified using the design review method to verify that the methodology, design inputs, assumptions, computations, and results of this analysis are technically accurate, adequate, complete and in accordance with Regulatory Guide 1.82. The design review was performed in accordance with NEDP-5, Document Design Review.

The Excel spreadsheets were reviewed to confirm that the plotted results and calculated NPSH/containment pressure margins for the different accident and operational transient events are reasonable compared to the inputs. The calculation provides adequate explanations and justifications. Therefore, the reviewer finds this calculation to be acceptable for its intended safety related purpose

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Subject:				
TRANSIENT NF	SH/CONTAINMENT PRESSURE EV	ALUATION	I OF RHR AND CORE SP	RAY PUMPS

1.0 Purpose:

The purpose of this calculation is to determine the Net Positive Suction Head (NPSH) available to the Core Spray (CS) and Residual Heat Removal (RHR) pumps as a function of time after postulated accident and operational transient events in accordance with Regulatory Guide (RG) 1.82. The calculation specifically addresses the recirculation pump suction DBA-LOCA, Station Blackout (SBO), Appendix R (APP R), and Anticipated Transient Without Scram (ATWS) events.

The available NPSH is compared to the required NPSH for the respective pumps to demonstrate that adequate margins exist to ensure that the RHR and CS pumps perform their intended design safety functions. The containment pressure necessary to preclude pump cavitation is also determined. This calculation evaluates maximum pump flow rates, operation of drywell coolers, and containment sprays with minimum or maximum cooling water temperature and provides graphical representations of the sequences to support responses to Round 6 Requests for Additional Information (RAI) relative to BFN Units 1, 2 and 3 Extended Power Uprate (EPU) license amendment requests (TS-418 and TS-431).

2.0 References:

- 2.1 TVA Calculation MDQ0999970046, Revision 10
- 2.2 GE-ER1-AEP-06-334, GE Responses to NRC Request for Additional Information ACVB-37 and Draft TVA Letter, W79-060803-001
- 2.3 Sulzer Pumps (US) Inc. Document No: E12.5.1267 Rev 0, NPSH Transient Review RHR and Core Spray Pumps, 7/11/2006
- 2.4 TVA Calculation MDQ0023980143, Revision 2 (for RHR HX K-factor method)
- 2.5 TVA Vendor Datasheet for Aerofin Drywell Coolers, VPF2647-10-1 (see Attachment 2)
- 2.6 PROTOHX Version 4.00 QA software for heat exchanger performance analysis
- 2.7 Browns Ferry Nuclear Plant (BFN) Units 2 And 3 Proposed Technical Specifications (TS) Change TS - 418 - Request For License Amendment Extended Power Uprate (EPU) Operation ***
- 2.8 Browns Ferry Nuclear Plant (BFN) Unit 1- Proposed Technical Specifications (TS) Change TS - 431 - Request For License Amendment - Extended Power Uprate (EPU) Operation ***
- 2.9 NRC Requests for Additional Information for EPU RAI 6, June 26, 2006 Unit 1 and Units 2 and 3 letters from Eva A. Brown to Karl W. Singer ***
- 2.10 Heat Exchanger Specification Sheet, Perfex Corporation, vendor manual VTM-P160-0010 (VTD-P160-0030, R6)
- 2.11 TVA Calculation MDQ0064920353, Revision 1
- 2.12 C1320503-6924, Revision 2, BFN EPU Containment Overpressure (COP) Credit Risk Assessment
- 2.13 TVA Calculation NDQ0999920116, Revision 20, Appendix R Manual Action Requirements
- *** Information Only reference, not specifically cited in calculation for design input

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TRANSIENT NPSH/CONTAINMENT PRESSURE EVALUATION OF RHR AND CORE SPRAY PUMPS

- 2.14 EPU FTR T0400 R1, Containment System Response
- 2.15 Browns Ferry EOI-1, RPV Control, R11 (Unit 2), R8 (Unit 3)
- 2.16 Browns Ferry EOI-2, Primary Containment Control, R9 (Unit 2), R7 (Unit 3)
- 2.17 EPU FTR T0611 R0, Appendix R Fire Protection
- 2.18 EPU FTR T0903 R0, Station Blackout
- 2.19 GE-ER1-AEP-06-340, W79-060811-002

3.0 Design Input Data:

3.1 Pump Flow Rates - Maximum flow rates per pump are determined from Ref 2.1 as follows:

Short Term post DBA-LOCA

RHR pump flow rate to the broken recirc loop = RHR pump flow rate to the intact recirc loop = CS pump flow rate =	11,500 gpm 10,500 gpm 4125 gpm
Long Term post DBA-LOCA	
RHR pump flow rate = CS pump flow rate =	6500 gpm 3125 gpm
Station Blackout (SBO) RHR pump flow rate =	6500 gpm
Appendix R (APP R) RHR pump flow rate =	9000 gpm
Anticipated Transient Without Scram (ATWS) RHR pump flow rate =	6500 gpm

- 3.2 Pump suction hydraulic losses and available NPSH without overpressure are determined for specified state point conditions from Tables 6, 10, and 13 of Ref. 2.1
- 3.3 Containment transient response parameters (suppression pool temperature, wetwell pressure, etc.) are obtained from Ref 2.2 and Ref 2.14
- 3.4 RHR and CS pump required NPSH as functions of flow rate and operating duration are obtained from the charts on pages 8 & 9 of Ref 2.3.
- 3.5 Initial suppression pool volume of 122,940 ft³ (TS Minimum with Drywell-to-Wetwell operating pressure differential) from MDQ0064920353, Rev. 1 (Ref. 2.11)
- 3.6 RHRSW maximum temperature of 92°F based upon highest recorded temperature during study for C1320503-6924, Rev. 2 (Ref 2.12)
- 3.7 RHR heat exchanger K value of 227 BTU/sec-°F per RHR heat exchanger based upon RHRSW temperature of 92°F (see Appendix A)
- 3.8 For other inputs used in the GE containment analyses, see Ref. 2.2



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4.0 **Assumptions:**

- For events involving containment spray cooling, the minimum service water temperature is 4.1 assumed to be 32F. Technical Justification: This limiting temperature condition conservatively maximizes the depressurization of the containment following containment spray cooling initiation.
- For events involving containment spray cooling (LOCA and SBO), the static head in the 4.2 suppression pool is reduced by the equivalent amount of water that would be required to flood the drywell floor holdup volume to the elevation of the downcomer pipe invert. Technical Justification: The drywell holdup volume will accumulate spillage from the break in the event of a LOCA and containment spray water following spray initiation after a LOCA or SBO. It is conservative for the NPSH computations to assume that this inventory is deducted from the initial pool inventory at the onset of the event scenario.
- 4.3 It is assumed that no makeup is provided from the condensate storage tank (CST). Technical Justification: Although the HPCI/RCIC systems would initially take suction from the CST to maintain reactor water level, compensating for coolant volume shrinkage during cooldown, this inventory of relatively cool water would reduce the pool temperature response and increase the pool level and pump suction static head. It is conservative to neglect this makeup source.
- UNVERIFIED ASSUMPTION see Section 6.2.3, Appendix R, Assumption 6. 4.4

5.0 **Requirements/Limiting Conditions:**

There are no operational requirements / limiting conditions for operation established by this analysis. The action sequences and timing are consistent with the current Emergency Operating Instructions and Technical Specifications relative to reactor and containment control and initiation of suppression pool cooling and containment sprays.

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TRANSIENT NPSH/CONTAINMENT PRESSURE EVALUATION OF RHR AND CORE SPRAY PUMPS

6.0 Computations and Analyses:

6.1 Methodology

The NPSH available is determined from the following standard equation for pumps with flooded suctions:

NPSHa = Hstatic + 144/ ρ (Pww - Pvapor) - HfWhere:Hstatic = water static head from the pool surface to the pump impeller centerline, ft
Pww = the containment wetwell pressure, psia
Pvapor = the water saturation pressure at the respective pool temperature, psia
 ρ = the density of the water at the respective pool temperature, lbm/ft³
Hf = the suction piping and strainer frictional head loss at the respective flow rate, ft

The available NPSH without credit for containment pressure is determined in Ref. 2.1 at specified pool temperature conditions (initial pump start, maximum pool temperature, and end of required overpressure credit period) and flow rates for each event. The TVA MultiFlow hydraulic flow balance software is employed in Ref. 2.1 to determine the suction head loss, Hf including the strainer head loss reflecting the appropriate debris loading for cases subject to post-accident debris generation (LOCA).

GE determined the containment response for each event using the Browns Ferry SHEX model (Ref 2.2) which provides the containment pressure and temperature transient conditions. To maximize suppression pool temperature and minimize containment pressure the mechanistic, non-equilibrium model of the mass and energy exchange between the pool surface and the wetwell atmosphere is applied in the subject NPSH analyses.

Sulzer Pumps evaluated the Browns Ferry RHR and CS pumps and provided charts of the required pump NPSH as a function of the flow rate and operational period in Ref 2.3 (see Attachment 1). The values for required NPSH are obtained from these charts by interpolation when necessary. Required NPSH values are selected for operating periods which bound the specific transient characteristic. For example, the APP R event is analyzed with one RHR pump operating at 9000 gpm and the following required NPSH values are applied:

RHR Flow Rate	Operating Time	NPSHr
gpm	hours	ft
9000	0-8	22.4-24.7
9000	8-12	24.7-25.6
9000	12-24	25.6-26.9
9000	24-100	26.9-34.6

For the APP R event, the NPSH requirement is incorporated as a continuous time variant function. For all other cases, the reqirement is applied as a step function for bounding operating periods.

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These parameters are input into a Microsoft ® EXCEL spreadsheet to calculate the available NPSH versus time for each event scenario. The steady-state NPSHa value from Ref. 2.1 is adjusted in the spreadsheet by replacing the steady-state vapor pressure and wetwell pressure with the transient vapor pressure and wetwell pressure, Pww. Available NPSH declines with increasing temperature. Therefore, the steady-state hydraulic calculations performed for the peak suppression pool temperature conditions are the base values in this analysis. No adjustment is made to the frictional and static head terms for the minor increase in fluid specific gravity at lower temperature conditions. The resulting transient available NPSHa is then compared to the required NPSHr to determine the margin available. Finally, the minimum containment pressure necessary to preclude pump cavitation is determined.

6.2 Analysis

This calculation analyzes the following event sequences to determine the NPSH available to the RHR and CS pumps as a function of time after the respective events:

- Loss of Coolant Accident (LOCA) Short Term
- Loss of Coolant Accident (LOCA) Long Term
- Anticipated Transient Without Scram (ATWS)
- Appendix R
- Station Blackout (SBO)

For each scenario, input parameters for this calculation are chosen from appropriate containment pressure/temperature models, flow models, available NPSH values and vendor supplied required NPSH information.

The spreadsheets for each event are documented in Appendix C and are identified as follows:

- EPU_RAI_6_LOCA.xls
- EPU_RAI_6_ATWS.xls
- EPU_RAI_6_APPR.xls and EPU_RAI_6_APPR_GE_R1.xls
- EPU_RAI_6_SBO.xls

Pertinent parameters (i.e. suppression pool temperatures, containment pressures and subject pump required containment pressures) are plotted to provide graphic representations of the time history of these events. The graphs for each case are included in Section 7 of this calculation.

The event-specific boundary conditions and assumptions are described in the following sections.



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TRANSIENT NPSH/CONTAINMENT PRESSURE EVALUATION OF RHR AND CORE SPRAY PUMPS

6.2.1 Loss of Coolant Accident (LOCA) - Short Term (ST)

The limiting design basis loss of coolant accident (LOCA), instantaneous double-ended rupture of one of the recirculation pump suction lines is postulated. The containment response is predicted with the SHEX code based on the following inputs and assumptions:

LOCA Containment Analysis Key Inputs

Item	Parameter	Value
1.	Reactor Power 102% of EPU power	4031 MWt
2.	Reactor Steam Dome Pressure	1055 psia
3.	Decay Heat	Decay heat used in the SHEX analysis is based on 102% of ANS 5.1-1979 decay heat with 2-sigma uncertainty adder
4.	Initial Suppression Pool volume corresponding to minimum suppression pool level	121,500 ft ³
5.	Initial Drywell Volume	159,000 ft ³
6.	Initial Wetwell Airspace Volume	129,300 ft ³
7.	Initial Drywell Pressure	15.5 psia
8.	Initial Drywell Temperature	150°F
9.	Initial Drywell Relative Humidity	100%
10.	Initial Wetwell Pressure	14.4 psia
11.	Initial Wetwell Temperature	95°F
12.	Initial Suppression Pool Temperature	95°F
13.	Initial Wetwell Relative Humidity	100%
14.	Ultimate Heat Sink/RHR Service Water Temperature	95°F
15.	RHR Heat Exchanger (HX) K value (per loop)	223 Btu/sec-°F
16.	Number of RHR Loops (1 RHR pump & 1 RHR HX per RHR loop)	4
17.	RHR Mode of Operation	LPCI and Pool Cooling
18.	Number of Drywell Coolers	0 (unavailable following LOOP)
19.	Heat Loads Modeled	Yes
20.	Heat Sinks in Drywell, Wetwell and Suppression Pool Modeled	Yes
21.	Leakage from the primary containment	2%/day

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TRANSIENT NPSH/CONTAINMENT PRESSURE EVALUATION OF RHR AND CORE SPRAY PUMPS

Pertinent Equipment Status:

- Two RHR Pumps at 11,500 gpm each (broken loop)
- Two RHR Pumps at 10,500 gpm each (intact loop)
- Four Core Spray Pumps at 4,125 gpm each
- No Containment Sprays in short term

LOCA-ST Assumptions:

- 1. The suppression pool is assumed to be initially at minimum technical specification level. **Justification** This minimizes the static head contribution to the available NPSH.
- 2. The suppression pool is assumed to be initially at maximum technical specification temperature. **Justification** This maximizes the temperature transient which minimizes the available NPSH.
- 3. Maximum drywell relative humidity and temperature are assumed. **Justification -** This minimizes the initial mass of non-condensable nitrogen in the containment and thus minimizes the transient pressure response.
- 4. Pumps start automatically and operate with wide open discharge valves with no operator intervention. Justification No operator actions are credited within the first ten minutes.
- 5. Pumps operate on their pump curves above their design flow rates (no throttling). Justification NPSH required increases and NPSH available decreases with flow rate so it is conservative to consider maximum flow rates.
- 6. Maximum suction strainer pressure drop is assumed, consistent with maximum debris loading. Justification - Maximum pressure drop is conservative for prediction of minimum NPSH available.

The time-history graphs of NPSH, containment pressure, and suppression pool temperature are presented in Section 7.

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Loss of Coolant Accident (LOCA) - Long Term

The LOCA-LT is the continuation of the LOCA-ST scenario from 10 minutes until suppression pool temperature is reduced and containment overpressure credit is no longer required for adequate NPSH. The limiting long-term LOCA scenario assumes loss of offsite power and single active failure of one division of emergency AC power, providing one train of safety equipment for accident mitigation.

Pertinent Equipment Status:

- Two RHR Pumps at 6,500 gpm each
- Two Core Spray Pumps at 3125 gpm each
- Containment Spray cooling mode initiated at 10 minutes

LOCA-LT Assumptions:

- Pumps are assumed to operate at their design flow rates under operator control. Justification- Emergency Operating Instruction (EOI) entry conditions for initiation of containment sprays are satisfied for this event and the operators will respond accordingly.
- RHRSW is assumed to be supplied to the RHR heat exchangers at either 32°F or 95°F (two
 extremes are analyzed to determine the limiting case). Justification Cold cooling water
 minimizes the containment spray temperature and produces the most rapid reduction in
 containment overpressure. Maximum cooling water temperature produces the maximum pool
 temperature response.

The time-history graphs of NPSH, containment pressure, and suppression pool temperature are presented in Section 7.



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6.2.2 Anticipated Transient Without Scram (ATWS)

For Browns Ferry, the limiting ATWS events are the Main Steam Isolation Valve Closure (MSIVC) and Pressure Regulator Failure-Open (PRFO). The containment response for these events is very similar, therefore the MSIVC event is selected for the NPSH evaluation.

ATWS Containment Analysis Key Inputs

Item	Parameter	Value
1.	Reactor Power 100% of EPU power	3952 MWt
2.	Reactor Steam Dome Pressure	1050 psia
3.	Decay Heat	Decay heat prior to reactor depressurization used in the ODYN analysis is based on the May-Witt model. Decay heat used in the SHEX analysis after reactor depressurization is initiated is based on nominal ANS 5.1-1979 decay heat (i.e., with no uncertainty adder).
4.	Initial Suppression Pool volume corresponding to minimum suppression pool level	122,940 ft ³
5.	Initial Drywell Volume	171,000 ft ³
6.	Initial Wetwell Airspace Volume	127,860 ft ³
7.	Initial Drywell Pressure	15.5 psia
8.	Initial Drywell Temperature	150°F
9.	Initial Drywell Relative Humidity	50%
10.	Initial Wetwell Pressure	14.4 psia
11.	Initial Wetwell Temperature	95°F
12.	Initial Wetwell Relative Humidity	100%
13.	Initial Suppression Pool Temperature	95°F
14.	Ultimate Heat Sink/RHR Service Water Temperature	92°F
15.	RHR Heat Exchanger (HX) K value (per loop)	227 Btu/sec-°F
16.	Number of RHR Loops (1 RHR pump & 1 RHR HX per RHR loop)	4
17.	RHR Mode of Operation	Pool Cooling
18.	Number of Drywell Coolers	10
19.	Heat Loads Modeled	Yes
20.	Heat Sinks in Drywell, Wetwell and Suppression Pool Modeled	Yes
21.	Leakage from the primary containment	2%/day

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Subject:

TRANSIENT NPSH/CONTAINMENT PRESSURE EVALUATION OF RHR AND CORE SPRAY PUMPS

Pertinent Equipment Status:

- Four RHR Pumps at 6,500 gpm each in Suppression Pool Cooling Mode
- All ten drywell air coolers remain in service

ATWS Assumptions:

- 1. Initial Drywell Relative Humidity is 50%. Justification Drywell RH normally ranges from 20% to 40% without coolant system leakage. The maximum value of 50% RH is selected to minimize the mass of non-condensable nitrogen in the drywell and thereby minimize the containment pressure response.
- 2. The operator initiates the Automatic Depressurization System (ADS) at the Heat Capacity Temperature Limit (HCTL) at approximately 20 minutes. **Justification** - The suppression pool temperature reaches the EOI-2 HCTL before reactor shutdown and RCS depressurization is required by.EOI-2 (SP/T-7).
- 3. Assume the operator uses the FW system to maintain water level after depressurization to replace HPCI when below HPCI isolation pressure. Justification EOI-1 (RC/L-4)
- 4. Both RHR trains (2 RHR pumps and HXs per train) are aligned in pool cooling mode. **Justification** This is consistent with EOI-2 (SP/T-7)
- 5. The drywell coolers (and drywell heat loads) are modeled. It is assumed that all 10 drywell coolers are operating. Justification Operating all 10 coolers minimizes containment pressure.
- 6. There is no leakage from the primary system to the drywell. Justification This assumption minimizes drywell pressure.

The ATWS containment response analysis is conducted in two parts corresponding to the period from event initiation until reactor depressurization and the period subsequent to depressurization. The first phase of the transient is modeled with the Browns Ferry ODYN model which determines the reactor power response and MSRV flow which is input to the SHEX containment model to determine the initial suppression pool temperature increase to the HCTL (180F). At that point the RCS is depressurized and , the SHEX containment code with a shutdown power curve is utilized to determine the long term, post-depressurization response.

Time-history graphs of NPSH, containment pressure, and suppression pool temperature are presented in Section 7.



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6.2.3 Appendix R (Fire Safe Shutdown)

The limiting APP R event for containment response, previously identified and analyzed in EPU Task Report 0611 (Ref. 2.17) as APP R Case 1 postulates the following:

- No spurious operation of plant equipment.
- Depressurization begins at 25 minutes using three main steam relief valves (MSRVs).
- One RHR pump aligned in the Low Pressure Coolant Injection (LPCI) mode, one RHR heat exchanger, and one RHR service water (RHRSW) pump is initiated at 2 hours

GE re-analyzed this event with their approved SHEX containment code utilizing the following inputs and assumptions:

Item	Parameter	Value
1.	Reactor Power 100% of EPU power	3952 MWt
2.	Reactor Steam Dome Pressure	1055 psia
3.	Decay Heat	Decay heat used in the SHEX analysis is based on nominal ANS 5.1-1979 decay heat (i.e., with no uncertainty adder).
4.	Initial Suppression Pool volume corresponding to minimum suppression pool level	122,940 ft ³
5.	Initial Drywell Volume	171,000 ft ³
6.	Initial Wetwell Airspace Volume	127,860 ft ³
7.	Initial Drywell Pressure	15.5 psia
8.	Initial Drywell Temperature	150°F
9.	Initial Drywell Relative Humidity	50%
10.	Initial Wetwell Pressure	14.4 psia
11.	Initial Wetwell Temperature	95°F
12.	Initial Wetwell Relative Humidity	100%
13.	Initial Suppression Pool Temperature	95°F
14.	Ultimate Heat Sink/RHR Service Water Temperature	92°F
15.	RHR Heat Exchanger (HX) K value (per loop)	227 Btu/sec-°F
16.	Number of RHR Loops (1 RHR pump & 1 RHR HX per RHR loop)	1
17.	RHR Mode of Operation	9400 gpm in LPCI mode until RCS depressurization, then 6000 gpm in Alternate Shutdown Cooling mode
18.	Number of Drywell Coolers	10 for first 2 hours, then isolated
19.	Heat Loads Modeled	Yes

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20.	Heat Sinks in Drywell, Wetwell and Suppression Pool Modeled	Yes	
21.	Leakage from the primary containment	2%/day	

Pertinent Equipment Status:

- One RHR Pump in ASDC mode at 2 hours
- 6,000 gpm assumed for minimum heat removal (Ref. Attachment 2)
- 9,000 gpm assumed for maximum required NPSH
- Ten Drywell Coolers continue operation until isolated by operator action at 2 hours

APP R Assumptions:

- 1. Initial Drywell Relative Humidity is 50%. Justification Drywell RH normally ranges from 20% to 40% without coolant system leakage. The maximum value of 50% RH is selected to minimize the mass of non-condensable nitrogen in the drywell and thereby minimize the containment pressure response.
- 2. RHRSW Temperature of 92°F. Justification Based upon highest recorded temperature during study for C1320503-6924, Rev. 2 (Ref. 2.12)
- 3. RHR heat exchanger K value of 227 BTU/sec-°F per RHR heat exchanger. Justification Based upon using RHRSW Temperature of 92°F (see Appendix A).
- 4. Use initial suppression pool volume of 122,940 ft³ (TS Minimum with Drywell-to-Wetwell operating pressure differential). Justification MDQ0064920353, Rev. 1 (Ref. 2.11)
- 5. Pump is running on its pump curve above its design flow rate (no throttling). Justification NPSH required increases and NPSH available decreases with flow rate so it is conservative to consider maximum flow rates.
- 6. It is assumed that the unit operators isolate all drywell coolers at 2 hours after the start of the fire event based on the recognition that the reduction in drywell pressure by the coolers and the pool temperature increase due to the MSRV discharge are challenging the available NPSH to the RHR pump(s). Justification This is an UNVERIFIED ASSUMPTION which will be resolved with the subsequent revision of the Appendix R Manual Actions Requirements calculation (Ref. 2.13) which is listed as a successor document to this calculation in CCRIS.

Time-history graphs of NPSH, containment pressure, and suppression pool temperature are presented in Section 7.



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6.2.4 Station Blackout (SBO)

The Station Blackout event sequence defined in EPU Task Report 0903 was re-analyzed by GE with the approved SHEX code methodology. Inputs and assumptions consistent with RG 1.82 to maximize pool temperature and minimize containment pressure were applied as follows:

- Unit trips when SBO occurs with automatic initiation of RCIC/HPCI systems to provide initial level control.
- Initial pressure control by automatic MSRVs operation.
- Operator action taken at one hour to control depressurization by cycling MSRVs while level control is maintained automatically by RCIC system.
- Coping duration is 4 hours.

Item	Parameter	Value
1.	Reactor Power 100% of EPU power	3952 MWt
2.	Reactor Steam Dome Pressure	1055 psia
3.	Decay Heat	Decay heat used in the SHEX analysis is based on nominal ANS 5.1-1979 decay heat (i.e., with no uncertainty adder).
4.	Initial Suppression Pool volume corresponding to minimum suppression pool level	121,500 ft ³
5.	Initial Drywell Volume	171,000 ft ³
6.	Initial Wetwell Airspace Volume	129,300 ft ³
7.	Initial Drywell Pressure	15.5 psia
8.	Initial Drywell Temperature	150°F
9.	Initial Drywell Relative Humidity	100%
10.	Initial Wetwell Pressure	14.4 psia
11.	Initial Wetwell Temperature	95°F
	Initial Wetwell Relative Humidity	100%
12.	Initial Suppression Pool Temperature	95°F
13.	Ultimate Heat Sink/RHR Service Water Temperature	95°F and 32°F
14.	RHR Heat Exchanger (HX) K value (per loop)	223 Btu/sec-°F
15.	Number of RHR Loops (1 RHR pump & 1 RHR HX per RHR loop)	2
16.	RHR Mode of Operation	Containment Spray Cooling mode at 4 hours
17.	Number of Drywell Coolers	0, unavailable due to SBO
18.	Heat Loads Modeled	Yes
19.	Heat Sinks in Drywell, Wetwell and Suppression Pool Modeled	Yes
20.	Leakage from the primary containment	2%/day

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Pertinent Equipment Status:

- Two RHR Pumps initiated in Containment Spray Mode at 4 hours
- Containment heat removal is based on assumed 5850 gpm each pump (Ref 2.18)
- NPSH is evaluated assuming 6500 gpm each pump

Assumptions:

1. Containment spray and suppression pool cooling at 4 hours. Justification - The analysis demonstrates that the EOI entry conditions for containment spray are met at the end of the coping period when power is restored.

Two SBO cases are analyzed to demonstrate the sensitivity of the available NPSH to the extremes of spray water temperature (i.e., RHRSW water temperature).

Time-history graphs of NPSH, containment pressure, and suppression pool temperature are presented in Section 7.



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Table 6.2-1 - SUMMARY OF NPSH AND CONTAINMENT PRESSURE MARGINS

SUMMARY OF NPSH RESULTS								
CASE	PUMP	FLOW	MINIMUM NPSHA	REQUIRED NPSH	MINIMUM NPSH MARGIN	MAXIMUM REQUIRED CONTAINMENT PRESSURE	MINIMUM CONTAINMENT PRESSURE MARGIN	DURATION OF REQUIRED COP
		GPM	FT	FT	FT	PSIA	PSI	
	cs	4125	26.5	25.5	1.0	16.4	0.4	9 min.
LOCA-ST	RHR-IL	10500	29.4	25.5	3.9	15.2	1.6	5 min.
	RHR-BL	11500	26.4	28.4	2.0	17.7	-0.9	10 min.
LOCA-LT	CS	3125	35.1	24.5	10.6	12.6	4.5	0
SPRAYS, 32F RHRSW	RHR	_6500	38.5	23	15.6	9.8	6.6	0
LOCA-LT	<u>cs</u>	3125	36.3	29	7.3	17.4	3.1	22.5 hours
SPRAYS, 95F RHRSW	RHR	6500	39.8	23	16.8	13.4	7.1	0
ATWS ALL DW COOLERS	RHR	6500	24.3	21.5	2.8	16.3	1.2	1.2 hours
APP R ALL DW COOLERS	RHR	7200	11.8	24.1	-12.3	27.5	-6.8	NA
APP R NO DW COOLERS	RHR	7200	32.6	22.4	10.2	23.3	2.6	71 hours
APP R DW COOLERS FOR 2HRS	RHR	9000	26.9	23	3.9	24	1.6	69 hours
SBO SPRAYS, 32F RHRSW	RHR	6500	27.6	21.5	6.1	15.6	2.5	0.6 hour
SBO SPRAYS, 95F RHRSW	RHR	6500	32.2	21.5	10.7	15.8	4.5	1.4 hours

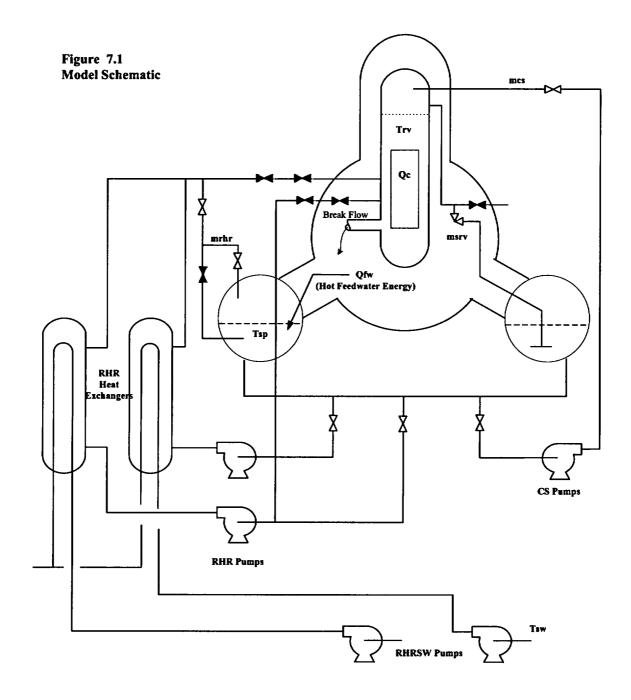
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7.0 Supporting Graphics:





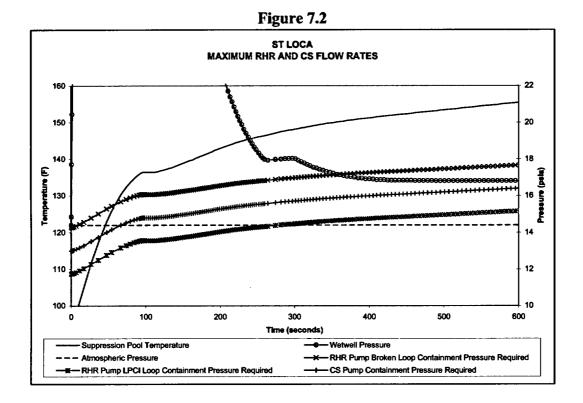
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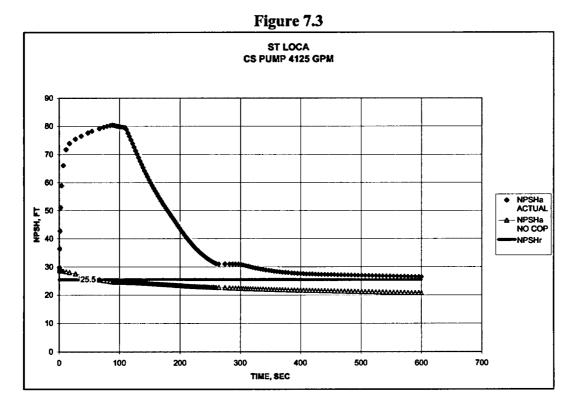
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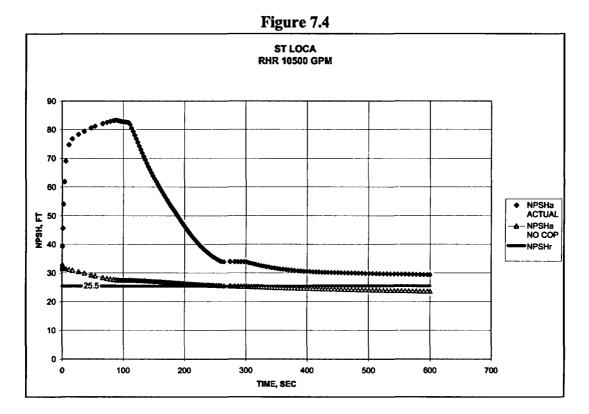
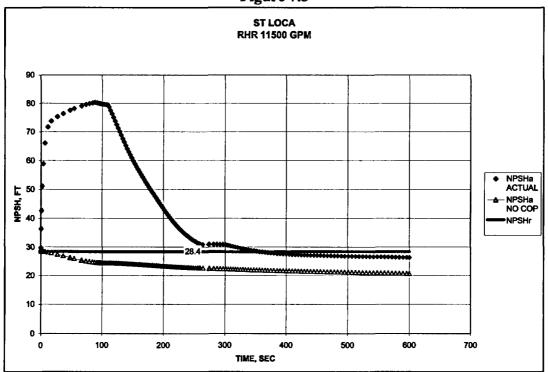


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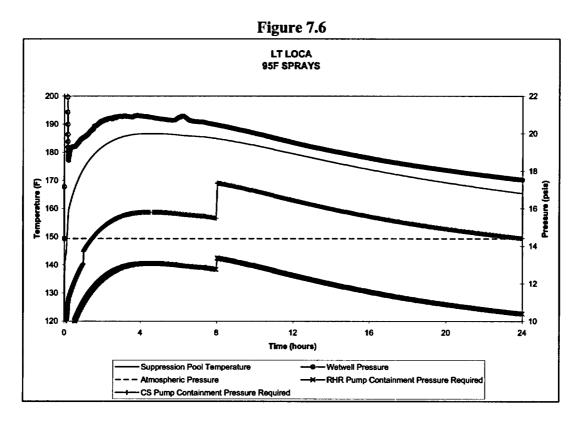


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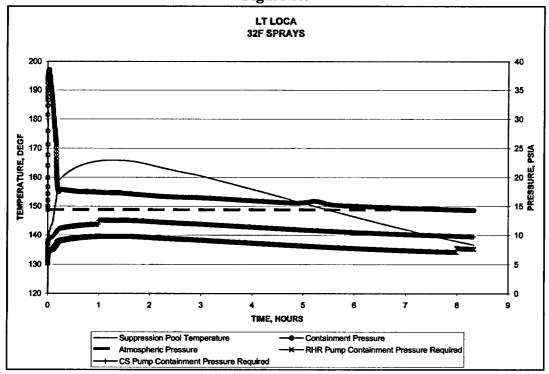
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Figure 7.8

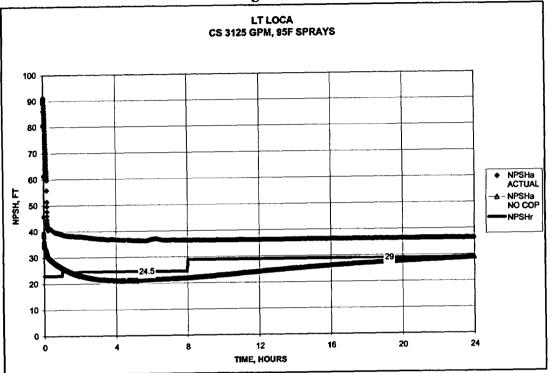
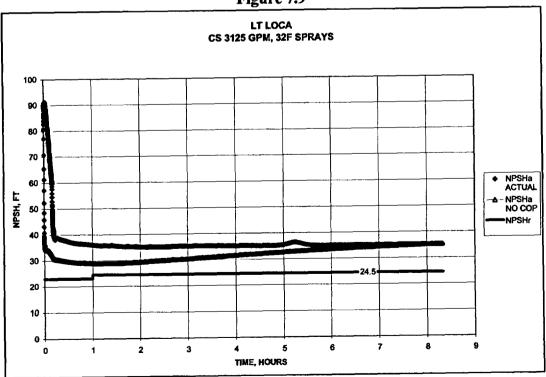


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Figure 7.10

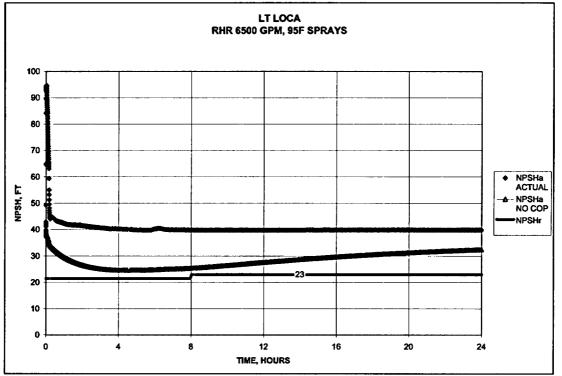
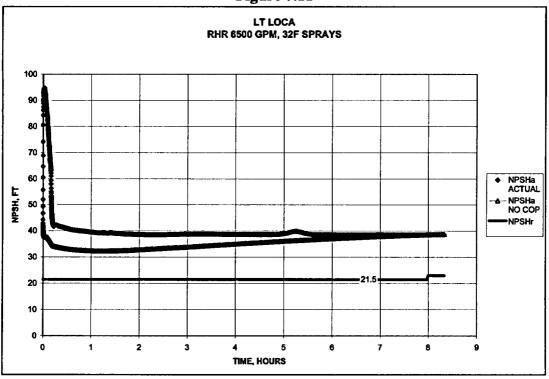


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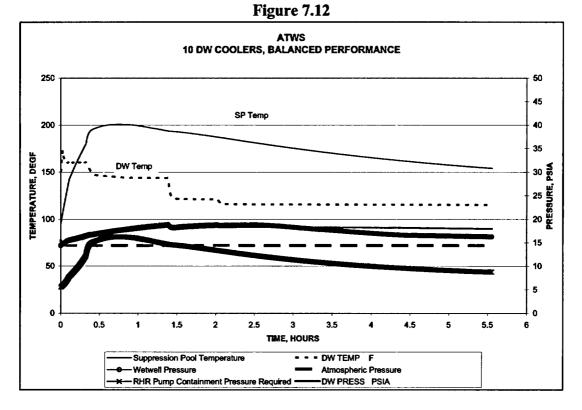
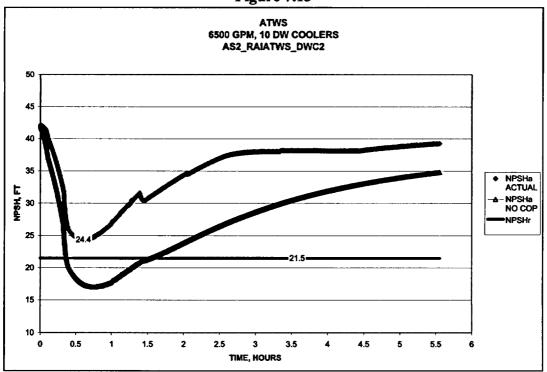


Figure 7.13



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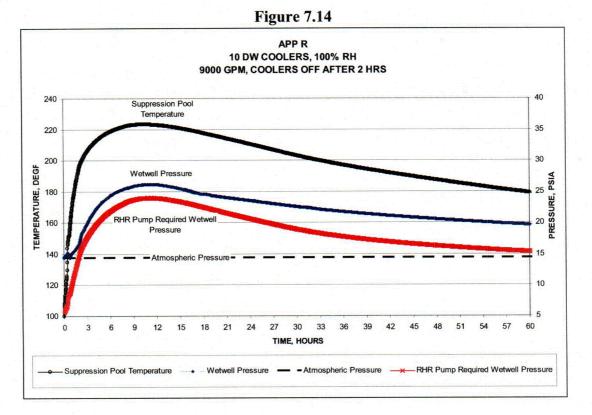
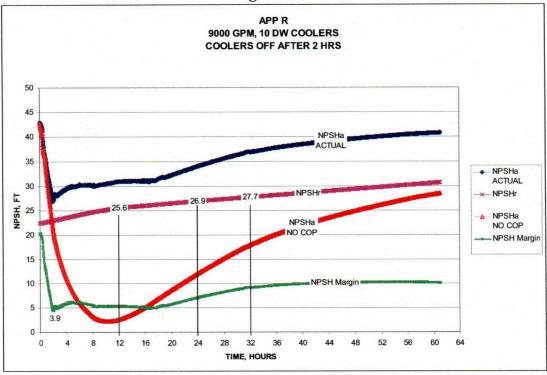


Figure 7.15





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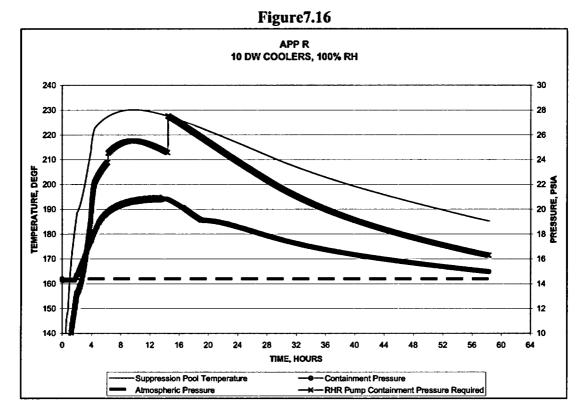
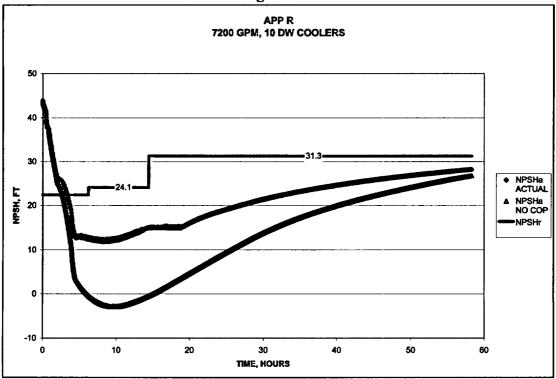


Figure7.17





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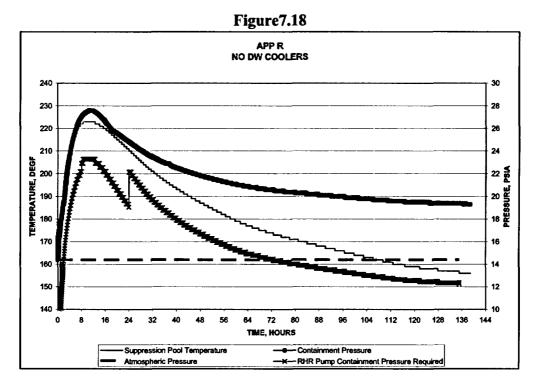
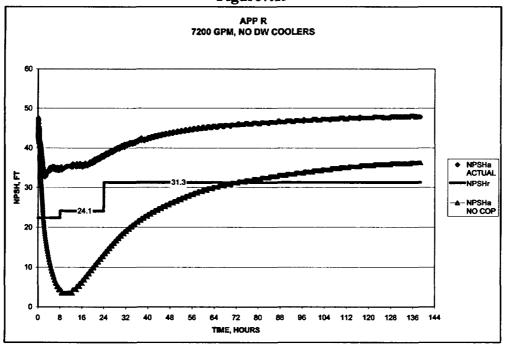


Figure7.19





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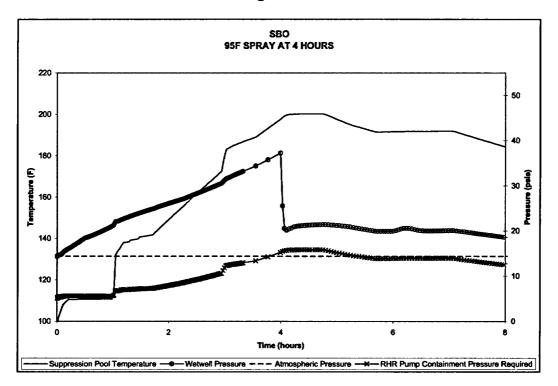
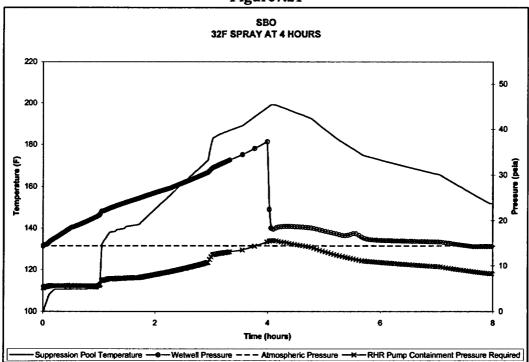


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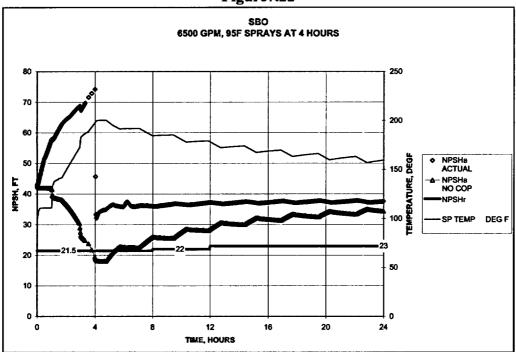
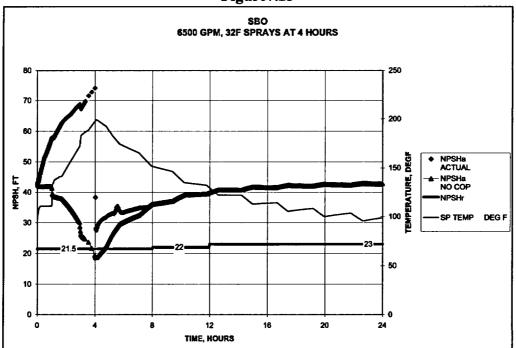


Figure7.22

Figure7.23





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8.0 Summary of Results:

The case parameters and results are presented in Table 6.2-1 (page 21) and in Figures 7.2 through 7.23.

9.0 Conclusions:

Analyses determining the Net Positive Suction Head (NPSH) available to the Core Spray (CS) and Residual Heat Removal (RHR) pumps as a function of time after postulated accident and operational transient events in accordance with Regulatory Guide (RG) 1.82 have been performed. The calculations are performed at EPU bounding conditions specifically for the recirculation pump suction DBA-LOCA, Anticipated Transient Without Scram (ATWS), Appendix R (APP R), and Station Blackout (SBO) events.

This calculation evaluates maximum pump flow rates, operation of drywell coolers, and containment sprays with minimum or maximum cooling water temperature and provides graphical representations of the sequences to support responses to Round 6 Requests for Additional Information (RAI) relative to BFN Units 1, 2 and 3 Extended Power Uprate (EPU) license amendment requests (TS-418 and TS-431).

Comparison of the available NPSH to the required NPSH for the respective pumps demonstrates that adequate margins exist to ensure that the RHR and CS pumps perform their intended design safety functions. The containment overpressure necessary to preclude pump cavitation and the duration for the required COP credit are determined for each event as summarized in Table 6.2-1 (page 21).

Specific conclusions are:

- The small deficiency (-2 ft) for the RHR pumps discharging to the broken loop in the LOCA-ST case is considered acceptable on the basis of the short (<10 min.) duration involved.
- Maximum spray temperature produces the maximum duration for the wetwell overpressure requirement as indicated by comparison of the two LOCA-LT cases for 32F and 95F RHRSW.
- The case which results in the minimum available NPSH margin is the ATWS event which reflects a minimum margin of 2.8 ft for a short duration of approximately one hour. This is a consequence of the conservative power input included in the ATWS model.
- A small margin is also predicted for the Appendix R event (3.9 ft). This event requires operator action to isolate the drywell coolers within the first two hours of the event scenario.
- The maximum duration for required wetwell overpressure is determined by the Appendix R event with coolers isolated at two hours.
- Continued operation of drywell coolers for events which do not involve LOCA break flow or drywell sprays (ATWS and Appendix R) minimizes the containment pressure and NPSH margins for those cases.



APPENDIX A

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TRANSIENT NPSH/CONTAINMENT PRESSURE EVALUATION OF RHR AND CORE SPRAY PUMPS

1.0 Purpose

The purpose of this Appendix is to calculate the heat exchanger K-factor per RHR heat exchanger required to transfer the design basis heat loads for the applicable PRA event sequences described in Section 6 of the main calculation. The result of this calculation serves as input to containment analysis that is used in the main calculation to provide time histories of the event sequences.

2.0 References

- 2.1 TVA Calculation MDQ0023980143, Revision 2 RHR Heat Exchanger Tube Plugging Analysis For Power Uprate (RIMS W78030630006)
- 2.2 Textbook entitled Fundamentals of Heat and Mass Transfer by Frank P. Incropera and David P. DeWitt, John Wiley & Sons, 3rd Edition
- 2.3 Power Uprate Evaluation Report for the Tennessee Valley Authority, Browns Ferry Units 2 and 3, Primary Containment System," General Electric Design Record File GE-NE-B13-01866-4, Rev 1, July 1998, (RIMS W79 980716 001)

3.0 Design Input Data

3.9 TVA Calculation MDQ0023980143, Revision 2 - RHR Heat Exchanger Tube Plugging Analysis For Power Uprate (RIMS W78030630006)

4.0 Documentation of Assumptions

- 4.1 RHRSW Temperature of 92°F. Justification Based upon highest recorded temperature during study for C1320503-6924, Rev. 2
- 4.2 Suppression Pool Temperature of 187.3°F. Justification reference 2.3.
- 4.3 RHRSW flow is 4000 gpm per pump. Justification reference 2.1
- 4.4 RHR flow is 6500 gpm per pump. Justification reference 2.1

5.0 Special Requirements/Limiting Conditions

None.

6.0 Computations and Analysis

6.1 Methodology

This Appendix uses an RHR heat exchanger model developed for reference 2.1. This model is an Excel spreadsheet application that uses known heat exchanger parameters and accepted standard engineering formulas to solve for unknown parameters. The accuracy of this model was confirmed previously (reference 2.1).

The major equation deals with heat exchanger effectiveness as a function of the overall heat transfer coefficient, the effective heat transfer area, the minimum mass flow-heat capacity product, and the heat capacity ratio. The effectiveness for a single shell pass, two tube pass CES type heat exchanger is given by reference 2.2, as follows:



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$$\varepsilon = 2 * \left[1 + C_{R+} \left[\left(1 + C_{R}^{2} \right)^{0.5} \right] \cdot \left[\frac{1 + e^{-NTU} \left[\left(1 + C_{R}^{2} \right)^{0.5} \right]}{1 - e^{-NTU} \left[\left(1 + C_{R}^{2} \right)^{0.5} \right]} \right] \right]^{-1}$$
(Eqn 1)

where:

3	= heat exchanger effectiveness
C _R	= heat capacity ratio
	= Cmin/Cmax
Cmin	= minimum mass flow rate times fluid heat capacity product, Btu/hr-°F
Cmax	= maximum mass flow rate times fluid heat capacity product, Btu/hr-°F
NTU	= number of transfer units
	= UA/Cmin
U	= overall heat transfer coefficient, Btu/hr-ft ^{2_o} F
^	- effective heat transfer area, the

A = effective heat transfer area, ft^2

This equation along with others shown in reference 2.1, were programmed in an Excel spreadsheet and solved in the following sequence: (Note all equation references come from reference 2.1)

- 1. To determine the heat exchanger performance at any flow condition, the inside and outside fluid film resistance terms are calculated with Eqns 3 & 4 respectively.
- 2. The new overall heat transfer coefficient, U is determined from Eqn 2.
- 3. The effective heat transfer area is found from Eqn 5 for the assumed tube plugging percentage (for this appendix 1.5% is used).
- 4. The mass flow rates and heat capacity rates are found from Eqns 6 & 7.
- 5. The effectiveness is determined from Eqn 1
- 6. From the effectiveness and minimum heat capacity rate, the K-factor is calculated.

The Excel spreadsheet is presented in Table 1.

6.2 Analysis

Using the accepted model, the following parameters were evaluated:

RHRSW Temperature = 92°F. Suppression Pool Temperature = 187.3°F RHRSW flow = 4000 gpm per pump. RHR flow = 6500 gpm per pump.

The spreadsheet solves for heat exchanger K-factor as indicated in Table 1.



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able 1			ļ	1											[· · · · · ·····		1	
(&Qasf	unction of R	HR Temp.	i at fixed TP9	6 and RHRS	W condition	18		-						/ / /	•		1 		
		4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	CONTAIN	MENT SPRA	Y DESIGN	CASE		-	-			Shell rhr	Tube sw				1	i <u></u>	
	1		1	Shell rhr	Tube sw						fluid res	0.000515	0.000857				t in the second		
			Tinlet	165	90		1				fouling res	0.0005	0.0023	•	1			•	1
			vsubf @ 6	0.016043	0.016043						metal res		0.000473	• • • • • • • •				• • • • • • • • • • • • • • • • • • •	
			Flow, gpm	10000	4500						total res		0.004645	1					1
	1		lbm/sec	1388.913	625.0109						U overali		215.285253						
																			ļ
					6500	GPM									• • • •			 	
		T			0-1-	0	40.1/4-1-10	1	D										
îmr =	RHR Flow	E	gpm	SW Flow	Cmin	Cmax	1/hi(do/di)	1/no	Rtotal	U	EPS	and and a set of the		RHRSW		Aeff	Q Dhullon	Tout_rhr	Tout_sw
120		95			553,1059	893,7808	0.000942	0.000667	0.004882	204.8519	0.404412	223,6827	40262891.4	total, gpm 8000		6107.34	Btu/hr 20131445.7	142 7424	105 110
130		95				891.4327	0.000942	0.000667	0.004882	204.8519	0.404412	223.5996	56347108.6	8000		6107.34	······		
140		95				888.9081	0.000942	0.000667	0.004882	204.8519	0.4041	223.5098	72417190.9		4.572807	6107.34			113.1845
150			4			886.217	0.000942	0.000667	0.004882		0.403926	223.4136	88471794.3	8000	4.572807	6107.34		136,1346	
160	883.3683	95	4000	553,1059	553,1059	883.3683	0.000942	0.000667	0.004882		0.40374	223.3112	104509632	8000	*	6107.34	52254815.8		
180	877.2294	95	4000	553,1059	553.1059	877.2294	0.000942	0.000667	0.004882	204.8519	0.403337	223.0883	136530069	8000		6107.34	68265034.4	158.3836	129.2837
187.3	874.817	95	4000	553.1059	553.1059	874.817	0.000942	0.000667	0.004882	204.8519	0.403178	223	148357440	8000	4.572807	6107.34		163.8463	132.2536
227	860.7125	92	4000	553.4209	553.4209	860.7125	0.000942	0.000667	0.004882	204.8519	0.400960	226.8245	220473396	8000	1.5	6304	140006600	404 4000	147.330



APPENDIX A

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TRANSIENT NPSH/CONTAINMENT PRESSURE EVALUATION OF RHR AND CORE SPRAY PUMPS

7.0 Supporting Graphics

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None.

8.0 Summary of Results

This calculation establishes that the heat exchanger K-factor per RHR heat exchanger that is required to transfer the design basis heat loads for the applicable event sequences is 227 BTU/sec-°F.

9.0 Conclusions

The heat exchanger K-factor derived by this Appendix was conservatively derived and is reasonable and expected with consideration with the inputs. The result of this calculation will serve as conservative input to containment analysis that will be used for main calculation.



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1.0 Purpose

The purpose of this Appendix is to calculate the heat load rate contribution of the reactor water recirculation pump motors to the containment. The result of this calculation serves as input to containment analysis that is used in the main calculation to provide time histories of the event sequences.

2.0 References

2.1 Cameron Hydraulic Data - 19th Edition.

3.0 Design Input Data

- 3.1 Rated horsepower of the reactor water recirculation pump motors is 8657 hp per page B3 of this Appendix.
- 3.2 The efficiency of the reactor water recirculation pump motors is 96.1% per page B4 of this Appendix.

4.0 Documentation of Assumptions

- 4.1 All horsepower lost due to inefficiencies is converted to heat that warms the drywell. Justification This will provide the most containment heat lost due to pump motor contribution at the time of the ATWS and Appendix R events to predict conservative drywell cooler performance.
- 4.2 Two Recirculation pumps are running at 100% rated flow. Justification This will provide the most containment heat lost due to pump motor contribution at the time of the ATWS and Appendix R events to predict conservative drywell cooler performance.

5.0 Special Requirements/Limiting Conditions

None.

6.0 Computations and Analysis

6.1 Methodology

This Appendix takes the rated horsepower of the reactor water recirculation pump motors and determines how much horsepower is lost due to inefficiencies. The lost horsepower is converted to heat rate using standard conversion factors.

6.2 Analysis

Efficiency loss = (100% - Efficiency)/100, thus

Efficiency loss = (100% - 96.1%)/100

Efficiency loss = 0.039

The per pump horsepower loss is given by

Horsepower loss/pump = (Efficiency loss) x (rated horsepower)

Horsepower loss/pump = (0.039) x (8657 hp) = 337.623 hp



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The horsepower lost for two pumps = 337.623 x 2 = 675.246 hp

Converting to a heat rate (Btu/sec):

(675.246 hp) x (42.43 Btu/min) x (1 min/60 sec) = 477.5 Btu/sec

7.0 Supporting Graphics

None.

8.0 Summary of Results

This calculation establishes that the heat load rate contribution to the containment for the reactor water recirculation pump motors is 477.5 Btu/sec.

9.0 Conclusions

The heat load rate contribution of the reactor recirculation pump motors as determined by Appendix was conservatively derived and is reasonable and expected with consideration with the inputs. The result of this calculation will provide the most containment heat lost due to pump motor contribution at the time of the ATWS and Appendix R events to predict conservative drywell cooler performance. The cooler performance will serve as conservative input to containment analysis that will be used for main calculation to provide time e histories of the event sequences.

10.0 Attachments

None



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Subject: TRANSIENT NPSH/CONTAINMENT PRESSURE EVALUATION OF RHR AND CORE SPRAY PUMPS

GE Motors	- DATA	SHEET -	GE Industrial Systems Custom 8000 (R) SQUIRREL CAGE MOTOR
CUSTOMER	: TVA BROWNS FERRY	//GE NUCLEAR	
CUSTOMER ORDER	<u>;: 00001704 01343 / 4310</u>	005962	
GE MODEL	: 291R610	DESIGN	: KC899R240B
SO	: 2880034 / 2880035	RI	: 132-1-6167 /02 /03
ατγ	: 1/1	SERIAL #	: 288000042 / 288000043
POWER	: 8657 HP	TYPE	: KV
	: 04		: 6800
VOLTAGE	: 3965 V		: ODP
	: 56.6 Hz		: 1.0 @ CLASS B RISE
PHASES	: 03		: F (POLYSEAL)
TEMPERATURE RISE	: 73°C/TC @ SF 1.0		:8
DRIVEN LOAD	: PUMP	MAX. ALTITUDE	: 3300 ft
LOAD WK2 REF. TO MOTOR SHAFT	: 12872 Lbft²	AMB. TEMP. (MIN/MAX)	:-18/57 °C
Calculated Performance	······································		
RATED RPM	: 1682	NEMA STARTING CODE	:E
	: 1066 A		: 570%
	: 26960 LbFt	LOCKED ROTOR TORQUE	: 60%
	: 7383		: 60%
	: Y		: 175%
	: 90% V		: FIXED (*)
	: CONT		: V1 - SOLID SHAFT
		ROTATION .	: CCW FROM TOP
TOTAL WEIGHT (calculated)	: 40000 Lb		
ROTOR WK2 (calculated)	: 15700 Lbft*	MAX. BRG.VIBR. [pk]	: .12 in/sec
·····,		BEARING TYPE	: SLEEVE (**)
		BRG LUBRIC(UPPER/LOWER)	
			: 0.01 in
STATOR RESIST. @ 25C	: 0.0139 Ohms L-L	LOCKED ROTOR TIME (100% V)	
-		COLD	: 20 s
X/R RATIO	: 38.700	(NEMA MG1-20.43) HOT	: 156
OPEN CIRC. TIME CONSTANT	: 2.6680 s		
		NUMBER OF STARTS	
	: M88D100134	വാ	
INSTRUCTION BOOK	: Later	НОТ	:1
		STATOR / ROTOR (SLOTS)	: 96/110
NOTES			
VIBRATION LIMITS BASED ON MOTO	R RUNNING UNCOUPLE	ED AND WITH BEARING TEMPERATURE STABL	LIZED IN STIFF BASE.
(*) PROVIDED BY CUSTOMER.			
(**) UP THRUST BAG - JV 15"			
(**) DOWN THRUST BRG - JV 13" 1/2"			
(**) UPPER SLEEVE GUIDE BRG - 8*			
(**) LOWER SLEEVE GUIDE BRG - 10			
SERVICE FACTOR @ CLASS F RISE			SH 1 0E 0
PREPARED BY :	OSVALDO AKIRA		SH 1 OF 2
APPROVED :	·		DS2880034
	06-30-2006 REV	:06	



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Ca	MODEL:	901DC1/						
		2011010)		SO: 2880034/2	880035	Ri : 132-1-6167 /	102 /03
121.42	lculated				3			
	No Load			159	5			
				4000				
	 75	0.91	96.1 96.0	1066 801	4			
	50	0.90	95.4	543]			
ACCES	SORIES				······································	TESTS		
1 VII 4 BR NOTES EQUI 9177 STAT TC # 1 # 2 . # 3 # 4	NVALENT 7 HP - 4200 TOR THEF SLOT P # 1 # 9 #17.	RATING: V - 1060 MOCOUP	OR - R. LE-COM IPLE LO TC #7 #8	87 rpm - 64 000000000000000000000000000000000000	IST-DOUBLE "T"	HIGH POTENTIAL TE INSULATION RESIST NO LOAD TEST NOISE POLARIZATION INDE POWER FACTOR TIF STATOR CORE TEST TORQUE & STARTIN WINDING RESISTAN PHASE SURGE TEST AC HIPOT (BEFORE SPRAY TEST SHAFT VOLTAGE LOCKED ROTOR TES ROTOR THERMAL S' VIBRATION ME BEARING INSULATIO SHAFT CRITICAL SP	AND BEARINGS (DUA ST ANCE TO GRD EX (BEFORE VPI AND P-UP TEST G CURRENT CE (BEFORE VPI; FINA VPI AND FINAL ASSY ST AT REDUCED VOL TABILITY TEST (COLL	IAL)) ; FINAL) TAGE D AND HOT T OWN)
[2] Acco [3] Acco [4] Acco [5] Acco	IONS ording to a ording to a ording to a ording to a ording to a	ustomer (ustomer (ustomer (ustomer (commer commer commer commer	nts. nts. nts. nts.				
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APPENDIX C

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TRANSIENT NPS	SH/CONTAINMENT PRESSURE EV/	ALUATION	OF RHR AND CORE SI	PRAY PUMPS

Excel Spreadsheet Computations

The spreadsheets used to develop the transient available NPSH curves and required containment pressure curves are setup with the GE transient containment parameters in the left hand columns, followed by three columns listing the atmospheric pressure, pool vapor pressure, and temperature dependent conversion factor for converting pressure to water head. The remaining columns to the right list the steady state NPSH case input in a block in rows 2-6 and the adjusted transient parameters below that for each pump considered for the subject event. An extract of the LOCA spreadsheet is presented on the following page and is typical of the other files.

The spreadsheet files identified as follows for each event are stored electronically as indicated on page 6:

- EPU_RAI_6_LOCA.xls
- EPU_RAI_6_ATWS.xls
- EPU_RAI_6_APPR.xls and EPU_RAI_6_APPR_GE_R1.xls
- EPU_RAI_6_SBO.xls



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Subject:			
TRANSIENT NPSH/CONTAINMENT PRESSURE	EVALUATION	OF RHR AND CORE S	PRAY PUMPS

	•	8	5	0	E	F	G	H	1	1 2 1	ĸ		-	N	0	•	<u> </u>	R			רט
Z		GE Comp	my Proprie	tary			DBA-LOC	A				-		LT LOCA C	ace 56	MHR 1C					
3										1				Pion gom	6500						
4		Buppleme	nt to OE-M	E-0009-006	8-2286-FD									Temp F	187.3						
5						H Evoluatio	•							Persy parts	14A		NPSHaft	CREDTO	1 2		
6		Ref. OE D/	ATAFRE	ef entil.	XLS									Pvep psta	8.812739		24.35	1			
7														MIN	MN	MiN	MN	MIN	MAX	M N	
8								155.93						33.5	34.7	323	21.5	17.0	2,2	7,2	
							WW					-									
1 1			OW		WW		PRESS								NPOM				Cont.	Cort.	1 1
	TIME		PRESS	DW TEMP	PRESS		2%LK	8P TEMP		PVAP 3P	Pelm			NPON	CREDIT	NPEHa		NPOH	Pressure	Pressure	1 1
9	HOURS	TIME	15 5C		PERA		PEIA 14.40	F		PSA	PSIA	Convitional		ACTUAL	1 23	NO COP	NPSH	Margin	Regit para	Margin pel	
11	1.93E-05	0.07	17.46	150.00	14 AC	95.01	14,40	95.00		0.815339		2 325392		42 907031				21 40703	5,17439	9,23	
	5.83E-C5	<u>0.0/</u> C.21	23 60	185.70			34,95		-	0.815339	14.4			42.907031	45.22742		21.5	21 40703 22 77353		9,23	
	0.0332-05	C.37	23.33	200.50	12.23		16.02	95.11	-	0.818098		2.323441			45.22146		21.5		5.177176	12,54	⊢−−1
	0.000145	0.52	25 54		17,16		17.16			0.820362		2.323482		49.300524					5.179461	11.98	<u> </u>
	0.000189	C.68	27.32	219.10	18.33	131.13	18.33			0.823136		2.320532		52 003742				33 50974		13.15	1
	0.000249	0.50	23.28				19.94			0.82744		2 325608		55 736851					5.186405	4.75	
	0.033336	1,21		236.00	22.03	155.90	22.03	95.78		0.835C81		2.120744			45.18478				5.194319	16.54	
48	0.000423	1.52	32.62	342.10	23.84	165.93	23.84	56.11		0.643558	\$4.4	2 323895	-	64754502	45,16645	42.84556	21.5		5.232875	18.54	
	0.033527	1.90	34.16	247.50	25.64	174.73	25.64	96.53		0.854454	34.4	2.321087		69 9 13913	45.14283	43.8218	21.5	47 A 1081	\$213472	23.A3	
23	137003.0	2,50	35.84	253.67	27.55	184.70	27.95			0.874746		2.321442		74.233348			21.5	5271305	5.234253	22.72	
	0.031031	3.71	39 02	246.73	30.73	195.30	35.73			0.916864		2.322167		B3 6C6448			21.5		5.276854	25.45	
	0.001378	4,95	33 57	254.83	32.32	201.00	32.32			0.966514		2.323202		84.204924		42.57673	21.5		5.226945	26.99	
	0.031725	6,21		267.13	33.25	204.35	11.5			1.016955		2.323331		85 273228					5.377824	27.57	
	0.000072	7.45		258.53	33.62	205.53	33.82	104.10	_	1.072827		2.324727		87 489299		42.34311			5.434165	28.39	
	0.032657	9,71	41.72	259.73	34.47 34.96	273.33	34.47 34.95	157.45		1.182166		2 3254 16		88.792358					5.544397	29.93	
	0.033292	14.71	42.16	270.53	36.94	211.70	35.39	111.E2 114.43		1.312277		2.328328		83 683534					5.675515	29.58	
	0.035405	19,45		270.93	36.15	217.93	35.39	120.73		1.725937		2.333199		90.426073 91.650081				73 15338		33.06	┢━━━┩
	0.035309	22.71		271.13	36.65	230.65	36.69			1.925575	14.4			52 512458		40.43766			6.293958	33,40	┢━━━┩
	0.031421	23.83	43.33	272.15	36.64		35.84	125.93		1.992532	44				42.63027			71,21512		33.48	
	0.005763	29.33	42.51	270.93	35.94		35.94	127.23	-	2.051851		2.337789	-		42,46995				6.433053	35.51	
	0.037534		43.93		37.05		37.55	128.93		2.159431	34.4			92.888514		39.91341			6.52718	33.52	
	0.0332*8	29.53	43 CE		37,15		37.15	130.53		2.252825		2.339582	-	92 931656		39.69934			6.622109	30.53	
34	0.035962	32.33	3976	290.90	37.24		37.24	131.93		2.338259	14.4	357646.2		92968761	41.54599			71 46876		33.53	
	0.00%21		39 54		37.29	223.65	37,24			2.394694	74.4	2 341372		92 970975	41,71833	33.37696	21.5	71 47095	6.764751	30.53	
	\$\$\$£03.0		39.41			223.70	37.31			2,43013		2 34 1634			41.66C71				6.790323	33,52	
	0.610162		3845	274.73	37.33		37.33			2.425524	14,4	2.3417		92 999696					6.796751	20.53	
	0.210159	36.57	39 05	255.90	17.33		37.33		_	2.425524	14,4	2.3417		92 999696					6,795751	30.53	
	0.010202	36.73	33 86	257.33	37.33		37.33			2.426524	54,4	2.3417			41.64622		21.5		6.796751	33.53	
43	0.013241	36,87	39.45	246,40	37.34		37.34	133.33	—	2425524	44	2,3417		93 023113					6,795751	33.54	
	0.012234	37.02	39.30	256.20	37,15	223.60	27.35		-	2.432933	14,4		_	93.033447	41.6317				6.803195	33.55	
	0.010358	37.29	39 32	256.10	37,35	223.90	<u>37.35</u> 37.37		<u> </u>	2.432533		2.341765		93 033447	41 6317	39,29993			6.803195	33,55	
	0.013757	38.73	397			223.53	37.40			2,4557 5		2.342028	-	93 097934			21.5		6.816125	33.55	
	0.011104		3378	256.90	37.44		37,44		┣—	2.475254		2342236		93 151713					6.845, 64	20.57	
	0.011265	40.91	39 33				27,47		-	2.497824		2343424		93.181571					6.865439	23,60	
	0.011426	41.85	39.24		37.50		37.50			2.513978				93,225225					6.881651	33.62	
	0.011268		39.45							2.524191	14.4			93 245325					6.894932	20.53	
					<u></u>		<u> </u>		L	4-34-121		Z	Law I	2992292			1 12	<u></u>	1.9.679744	<u> </u>	لسجيها

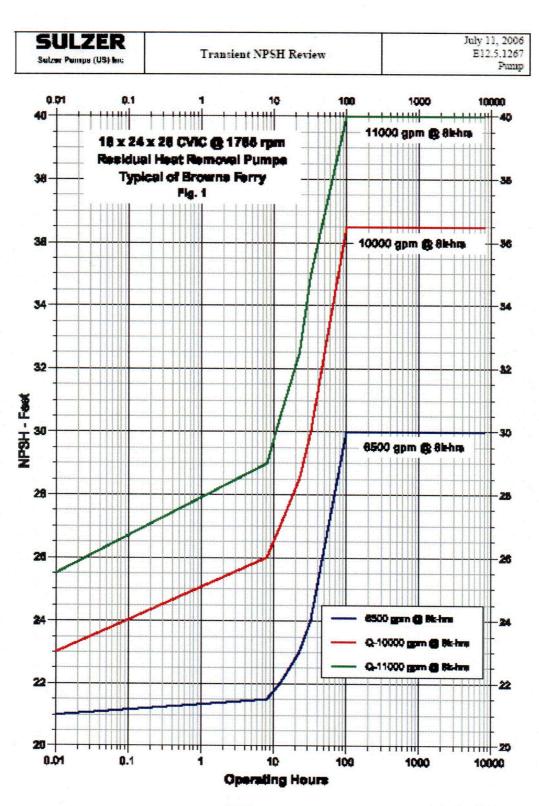


APPENDIX C

Calculation No.	MDQO99920060011	Rev: 1	Plant: BFN Unit 0	Page: C3 of 3
Subject:	SH/CONTAINMENT PRESSUR	E EVALUATION	OF RHR AND CORE S	

	1 1	ĸ	1	M	N	0	ρ	<u>م</u>	R	s	7	1 0
2				t	ILT LOCA Case 3B		RHR IC		······		· · · · · · · · · · · · · · · · · · ·	
3						6500				+		
4				1	Temp F	187.3	<u> </u>					
5						14.4		NPSHa t	CREDIT psq			
e				<u> </u>	Pvap psia	=pfie(O\$4)	<u> </u>	24.35	1	1		1
7					MiN	MiN	Min	MIN	Min	MAX	MON	ł
8				<u></u>	-MIN(N10,N1529)	=M:N(O10:01529)	=MIN(P10 P1529)	=MIN(Q10:Q1529)	=MIN(R10;R1529)		=MIN(T10;T1529)	<u> </u>
<u> </u>							-minter to the total		-waterouterouterouterouterouterouterouterou			
						NPSHa						
	PVAP SP				NPSHa	CREDIT	NPSHa			Cont. Pressure Regid	Cont. Pressure	
9	PSIA	Patm PSIA	Conv ft/psi		ACTUAL	1 PSI	NO COP	NPSHr	NPSH Maroin	01-2	Margin psi	
10	=pfte(H10)	14.4	=144"vftge(H10.0)	1	=\$Q\$0+(\$Q\$0-\$J10-\$C	=\$Q\$6+(\$Q\$6-\$J10-\$	-5056+(\$0\$6-\$J10-5		=N10-Q10	=14 4-(P10-Q10)110		1
11	=ofte(H11)	14.4	=144*vtoe(H111.0)	1	=3Q50+(SQ50-SU11-SC				=N11-Q11	=14 4-(P11-Q11)L11		t
12	=ofter(H12)	14.4	=144*vftge(H12.0)		=\$0\$6+(\$0\$6-\$112-\$0				=N12-Q12	=14.4-(P12-Q12)112		1
13	=pfterH13}	14.4	=144 vfige(H13,0)	i —	=\$Q\$6+(\$Q\$6-\$J13-\$C				-N13-013	14.4-(P13-G13)/L13		1
14	=pfte(H14)	14.4	=144"vftge(H114,0)	1	=5058+(\$058-\$J14-\$0				=N14-Q14	=14.4-(P14-Q14)/L14		î
16	=pfte(H15)	14.4	=144*vftge(H15,0)		=\$0\$0+(\$0\$0-\$J15-\$0				=N16-Q15	=14.4-(P15-Q15)1.15		Ì
10	=pite(H16)	14.4	=144'vftge(H10.0)		=\$0\$8+(\$0\$8-\$J18-\$0				=N16-Q18	=14.4-(P10-Q10)L10		1
17	=pfte(H17)	14.4	=144 vf:ge(H17,0)	1	=\$C\$6+(\$O\$6-\$J17-\$C				AN17-Q17	14.4-(P17-Q17VL17		1
18	=pfte(H18)	14.4	=144*sfige(H18,0)		=\$Q\$8+(\$Q\$8-\$J18-\$C	=\$Q\$0+(\$O\$0-\$J18-\$	=\$Q\$6+(\$Q\$6-\$J18-5	8(21.5	=N18-Q18	=14.4-(P18-Q18)L18	-\$318-\$18	
19	=pfte(H19)	14.4	=144"vftqe(H19,0)		=\$Q\$8+(\$Q\$8-\$J19-\$C	=\$Q\$6+(\$0\$6-\$J19-\$	=\$Q\$6+(\$Q\$6-\$J19-	\$ 21.5	-N19-Q19	=14.4-(P10-Q10)/L19	-\$319-519	
20 21	=pfte(H20)	14.4	=144*vfige(H20,0)		=\$Q\$8+(\$Q\$8-\$J20-\$C	=\$Q\$6+(\$0\$6-\$J20-\$	=\$Q\$8+(\$Q\$8-\$J20-5	\$ 21.5	=N20-Q20	=14.4-(P20-Q20)/L20		1
21	=pfte(H21)	14.4	=144 white(H21.0)		=\$Q\$8+(\$Q\$8-\$J21-\$C	=\$Q\$6+(\$0\$6-\$J21-\$	=\$Q\$0+(\$Q\$0-\$J21-	\$ 21.5	=N21-G21	=14.4-(P21-Q21)/L21	=\$021-921	1
22	=pite(H22)	14.4	=144 vt:qe(H22.0)		=\$Q\$6+(\$Q\$6-\$J22-\$C	=\$Q\$0+(\$O\$0-\$J22-\$	=\$Q\$8+(\$Q\$8-\$J22-	521.5	=N22-G22	= 14.4-(P22-Q22)/L22	=\$322-822	
23	=pfte(H23)	14.4	=144"vfige(H23.0)		=\$Q\$0+(\$Q\$0-\$J23-\$C	=\$Q\$0+(\$Q\$0-\$J23-\$	=\$Q\$0+(\$Q\$0-\$J23-\$	\$ 21.5	eN23-Q23	=14.4-(P23-Q23)/L23	=\$323-923	
24	mpfte(H24)	14.4	=144*vhqe(H24.0)		=\$Q\$0+(\$Q\$6-\$J24-\$C	=\$Q\$6+(\$Q\$6-\$J24-\$	=\$Q\$8+(\$Q\$6-\$J24-\$	\$21.5	*N24-Q24	=14.4-(P24-G24)/L24	=\$324-\$24	
26	=pfte(H25)	14.4	=144 vftqe(H25.0)		=\$Q\$0+(\$Q\$0-\$J25-\$C	=\$0\$6+(\$0\$6-\$J25-\$	=\$Q\$6+(\$Q\$6-\$J25-5	\$121.5	=N25-Q25	=14 4-(P25-G25)125	=9326-825	
20	=pfte(H26)	14.4	=144*vftge(H20,0)		=\$Q\$0+(\$Q\$6-\$J26-\$C	=\$Q\$6+(\$0\$6-\$J26-\$	=\$Q\$6+(\$Q\$6-\$J26-	\$121.5	=N26-Q26	=14 4-(P28-Q28)/L28	=9326-826	T
27	=pfte(H27)	14.4	=144*vftge(H27,0)		=\$Q\$8+(\$O\$8-\$J27-\$C				=N27-Q27	=14 4-(P27-Q27)127		
28 29	=pfte(H28)	14.4	=144*vftqe(H28.0)		=\$Q\$0+(\$Q\$0-\$J28-\$C	=\$Q\$6+(\$Q\$6-\$J28-\$	=\$Q\$0+(\$O\$0-\$J28-	\$ 21.5	*N28-Q28	=14.4-(P28-Q28)1_28		
29	=pfte(H29)	14,4	=144*vftqe(H29,0)		=\$G\$0+(\$O\$0-\$J29-\$C				=N29-Q29	=14.4-(P29-G29)129		
30	=pfte(H30)	54,4	=144 vftge(H30,0)		=\$0\$0+(\$0\$6-\$J30-\$C				=N30-Q30	=14 4-(P30-Q30)/L30		
31	=pfte(H31)	14.4	*144*vftqe(H31,0)		=\$Q\$6+(\$Q\$6-\$J31-\$C				=N31-Q31	=14 4-(P31-Q31)/L31		
32	=pfte(H32)	\$4.4	=144"vftqe(H32,0)	1.	=\$0\$8+(\$0\$8-\$J32-\$0				=N32-Q32	=14 4-(P32-Q32)/L32		
33	=pfte(H33)	14,4	=144*vftge(H33.0)		=\$Q\$6+(\$O\$6-\$J33-\$C				=N33-Q33	=14.4-(P33-Q33)/L33		
34	ep/te(H34)	14,4	=144*vitge(H34,0)		=\$\$\$6+(\$0\$6-\$J34-\$0				=N34-Q34	=14 4-(P34-Q34)/L34		
36	=pite(H35)	14,4	=144*vftge(H35,0)		=\$Q\$8+(\$O\$8-\$J35-\$C				=N35-Q35	=14 4-(P36-Q35)/L35		
30	=pfte(H36)	14.4	=144*vftqe(H38,0)		=\$Q\$8+(\$0\$8-\$J38-\$C				=N38-Q38	=14.4-(P38-Q38)/L38		L
37	=pfte(H37)	14.4	=144*vftqe(H37,0)		=\$Q\$8+(\$0\$8-\$J37-\$0				=N37-Q37	*14 4-(P37-Q37)/L37		
38	=pite(H32)	14.4	=144 vf:ge(H35.0)		=\$Q\$0+(\$O\$0-\$J35-\$C				=N38-Q38	=14 4-(P38-Q38)/L38		
39	=pfte(H39)	14.4	=144"vftge(H39.0)	L	=\$Q\$8+(\$0\$8-\$J39-\$C				=N39-Q39	=14 4-(P30-Q30)/L30		
40	=pfte(H40)	14,4	=144"vftge(H40.0)		=\$Q\$8+(\$0\$8-\$J40-\$0				=N40-Q40	=14.4-(P40-Q40)/L40		
41	=pfte(H41)	14.4	=144"vf:qe(H41.0)		=\$Q\$8+(\$Q\$8-\$J41-\$C				=N41-Q41	=14.4-(P41-Q41)/L41		L
42	=pfte(H42)	14.4	=144 vftge(H42,0)		=\$Q\$6+(\$0\$6-\$J42-\$0				=N42-Q42	=14.4-(P42-Q42)/L42		1
43	=pfte(H43)	14,4	=144 vftge(H43.0)		=\$Q\$0+(\$Q\$0-\$J43-\$C				=N43-Q43	=14 4-(P43-Q43)/L43		I
44	=pfte(H44)	14,4	=144 vftqe(H44.0)		=\$250+(\$050-5,)44-30				=N44-Q44	=14 4-(P44-Q44)/L44		<u> </u>
45	=pfte(H45)	14,4	=144*vftqe(H45.0)	1	=\$Q\$8+(\$0\$8-\$J45-\$0	<u>=\$Q\$6+(\$0\$6-\$J45-\$</u>	1=5Q\$0+(\$Q\$0-\$J45-	5(21.6	=N45-Q45	=14 4-(P45-Q45)/L45	=\$G45-\$45	

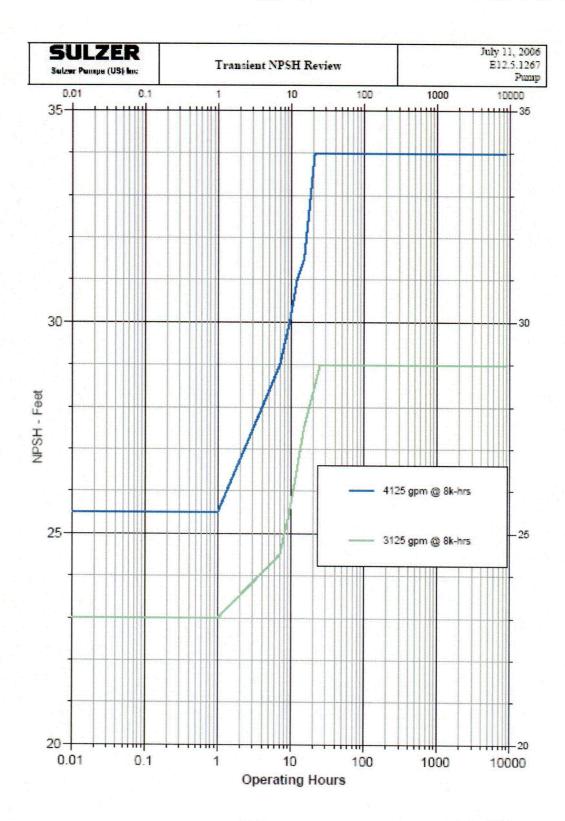
Calculation No.	MDQO99920060011	Rev: 0	Plant: BFN Unit 0	Page: 1 of 3



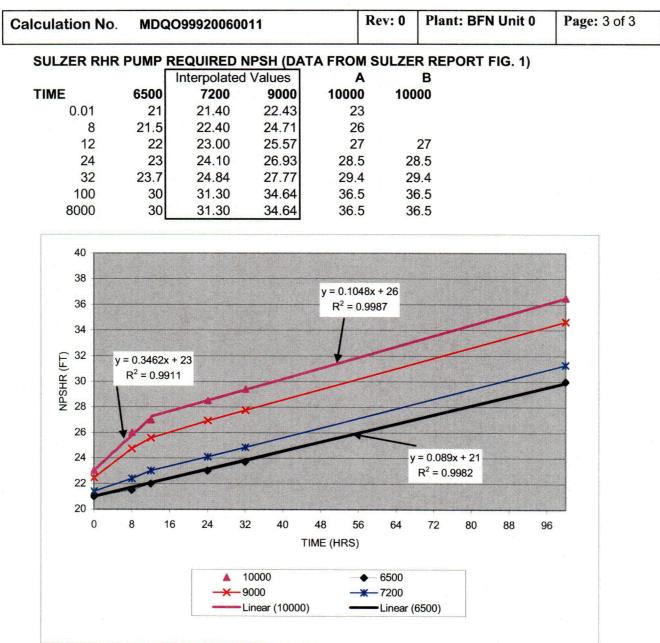
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DESIGN CONDITIONS

	4	UR .				F	RBCCW				LATENT		10 COILS	10 COILS	10 COILS	10 COILS
	F	LOW,	INLET	OULET		F	LOW,	RECCW	RBCCW	SENSIBLE	HT,	CONDENSATE.	SENSIBLE HT.	LATENT HT.	TOTAL HT.	CONDENSATE
CASE	A	CFM	T-08	T-DB	RH	0	SPM	IN	OUT	HT, BTU/HR	BTUHR	LBM/HR	BTU/SEC	BTUISEC	BTU/SEC	LBM/SEC
DESIGN		19000	144.5	111.5		٥	127.4	10	0 109,4	593036	i 0	0.00	1647	0	1647	
	4	19000	144.5	112.6		40	127.4	10	0 109.3	586669) 0	0.00	1630	0	1630	
	5	19000	144.5	113	- 19 A	43	127.4	10	0 109.4	581042	14318	14.76				
	1	19000	144.5	117.1		50	127.4	10	0 111.4	506636	228840	235,92	1407	636		
	3	19000	144.5	131.1		80	127.4	10	122.47	250094	1274385	1313.80				
	2	19000	144.5	138.3		100	127.4	10	0 132.11	114930	2102602	2167.84	319		6160	
	6	19000	133.7	108.1		50	127.4	. 9	5 104.3	520196	72029	74.26	1445	200		· · · · · · · · · · · · · · · · · · ·
	7	19000	133.7	126.9		100	127.4	9	5 121.8	137636	1690663	1742.95	382			
RAI-6.1		19000	150	115.23		40	127.4	10	0 110.99	670338	25438	26.22			1933	
RAI-6.2		19000	150	118.5		45	127.4	10	0 112.45	609065	195352	201.39			2234	
RAI-6.3		19000	150	121.53	F	50	127.4	10	0 114.16	552067	370265	381.72				

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	Calculation Report for BFN 150F, 50%RH, EPU			
	Calculatio	on Specifications	· · · · · · · · · · · · · · · · · · ·	

Constant Inlet Temperature Method Was Used Extrapolation Was to User Specified Conditions Design Fouling Factors Were Used

Test Data

Data Date Air Flow (acfm) Air Dry Bulb Temp In (°F) Air Dry Bulb Temp Out (°F) Relative Humidity In (°•) Relative Humidity Out (°•) Wet Bulb Temp In (°F) Wet Bulb Temp Out (°F) Atmospheric Pressure (psia) Tube Flow (gpm) Tube Temp In (°F) Tube Temp Out (°F) Condensate Temperature (°F)

Extrapolation Data						
Tube Flow (gpm)	127.40					
Air Flow (acfm)	19,000.00					
Tube Inlet Temp (°F)	100.00					
Air Inlet Temp (°F)	150.00					
Inlet Relative Humidity (%)	50.00					
Inlet Wet Bulb Temp (°F)	0.00					
Atmospheric Pressure (psia)	15.500					

*** Air Mass Velocity (Lhm hr ff), Tube Fluid Velocity (ft'sec); Air Density at Inlet T, Other Properties at Average T

Calculation No. MDQ09	9920060011	<u> </u>	Rev: 0	Plant: BFN Unit 0	Page: 3 of 7
	lculation Repo	by Proto-Power TV/ rt for BFN2CCL of RH, EPU RAI 6	- -070-740 - Dry		Page 2
	Extra	polation Calcu	lation Summ	Dary	
Mass Flow (lbm/hr) Inlet Temperature (°F) Outlet Temperature (°F) Inlet Specific Humidity Outlet Specific Humidity	Air-Side 68,850.34 150.00 121.53	Tube-Side 63,353.58 100.00 114.16	j Factor An-Side ho Tube Wall R	(BTU/br-ft ^{z.} °F) (BTU/br-ft ^{z.} °F) esistance (hr-ft ^{z.} °F/BTU) ing (hr-ft ^{z.} °F/BTU)	0.00 0.0000 0.00 0.00016250 0.00412058
			Effective An LMTD Total Heat T Surface Effe	ransferred (BTU/hr) ctiveness (Etz)	2,046.63 0.00 922,338 0.0000
			Latent Heat	it Transferred (BTU/hr) Fransferred (BTU/hr) lensate (BTU/hr)	552,067 370,265 26,889

Extrapolation Calculation for Row 1(Dry)								
	Air-Side	Tube-Side						
Mass Flow (lbm/hr)	68,850.34	63,353.58	Tube-Side hi (BTU/hr·ft².°F)	1,801.20				
Inlet Temperature (°F)	150.00	111.80	j Factor	0.0103				
Outlet Temperature (°F)	142.29	114.16	Air-Side ho (BTU/hr·ft ⁼ .°F)	15.46				
Inlet Specific Humidity	0.084838		Tube Wall Resistance (hr-ft ^z . °F/BTU)	0.00016250				
Outlet Specific Humidity	0.084838		Overall Fouling (hr ft ² °F/BTU)	0.00412058				
Average Temp (°F)	146.14	112.9785						
Skin Temperature (°F)	116.88	114.9949	U Overall (BTU/hr·ft [*] .°F)	13.29				
Velocity ***	5.047.79	6.2645	Effective Area (ft ⁼)	341.11				
Reynold's Number	1,849++	42,497	LMTD	33.01				
Prandti Number	0.7253	3.9144	Total Heat Transferred (BTU/hr)	149,618				
Bulk Visc (lbm/ft-hr)	0.0491	1.4407						
Skin Visc (Ibm/ft-hr)	0.0000	1.4125	Surface Effectiveness (Eta)	0.9743				
Density (lbm/ft*)	0.0612	61.817 2	Sensible Heat Transferred (BTU/hr)	149,618				
Cp (BŤŮ/lbm·°F)	0.2402	0.9988	Latent Heat Transferred (BTU/hr)	-				
K (BTU/hr ft °F)	0.0163	0.3676	Heat to Condensate (BTU/hr)					
Relative Humidity In (%)	50.00							
Relative Humidity Out (%)								

** Reynolds Number Outside Range of Equation Applicability

*** Air Mass Velocity (Lbm hr fr), Tube Finid Velocity (ft sec); Air Density at Inlet T, Other Properties at Average T

I GAICUIATION NO MUGU99970050011	Calculation No.	MDQQ99920060011
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TVA

Calculation Report for BFN2CCL-070-740 - Drywell Air Cooler

150F, 50%RH, EPU RAI 6 INITIAL CONDITION

Extrapolation Calculation for Row 2(Dry)

	Air-Side	Tube-Side	
Mass Flow (lbm/hr)	68,850.34	63,353.58	Tube-Side hi (BTU/hr-ft ^z .°F) 1,779.92
Inlet Temperature (°F)	142.29	109.79	j Factor 0.0103
Outlet Temperature (°F)	135.75	111.80	Air-Side ho (BTU/hr-ft ² .°F) 15.40
Inlet Specific Humidity	0.084838		Tube Wall Resistance (hr-ft ² .ºF/BTU, 0.00016250
Outlet Specific Humidity	0.084838		Overall Fouling (hr·ft ² ·°F/BTU) 0.00412058
Average Temp (°F)	139.02	110.7941	
Skin Temperature (°F)	114.12	112.5236	U Overall (BTU/hr·ft ² ·*F) 13.24
Velocity ***	5.047.79	6.2613	Effective Area (ft ²) 341.11
Reynold's Number	1.866++	41,586	LMTD 28.09
Prandtl Number	0.7259	4.0087	Total Heat Transferred (BTU/hr) 126,815
Bulk Visc (lbm/ft-hr)	0.0487	1.4723	
Skin Visc (lbm/ft-hr)	0.0000	1.4472	Surface Effectiveness (Eta) 0.9744
Density (lbm/ft')	0.0618	61.8487	Sensible Heat Transferred (BTU/hr) 126,815
Cp (BTU/lbm·°F)	0.2402	0.9988	Latent Heat Transferred (BTU/hr)
K (BTU/hr ft °F)	0.0161	0.3668	Heat to Condensate (BTU/hr)
Relative Humidity In (%)	60.67		
Relative Humidity Out (%)	71.82		

** Reynolds Number Outside Range of Equation Applicability

Extrapolation Calculation for Row 3(Dry)					
_	Air-Side	Tube-Side			
Mass Flow (lbm/hr)	68,850.34	63,353.58		761.82	
Inlet Temperature (°F)	135.75	108.09	j Factor	0.0102	
Outlet Temperature (°F)	130.20	109.79	Air-Side ho (BTU/hr ft ^z .°F)	15.35	
Inlet Specific Humidity	0.084838		Tube Wall Resistance (hr-ft ^z .ºF/BTU, 0.000	16250	
Outlet Specific Humidity	0.084838		Overall Fouling (hr·ft [*] ·°F/BTU) 0.004	12058	
Average Temp (°F)	132.97	108.9416	-		
Skin Temperature (°F)	111.78	110.4243	U Overall (BTU/hr-ft*-°F)	13.19	
Velocity ***	5.047.79	6.2587	Effective Area (ft [*])	341.11	
Reynold's Number	1.880++	40,819	LMTD	23.92	
Prandtl Number	0.7264	4.0917	Total Heat Transferred (BTU/hr) 10	07,615	
Bulk Visc (lbm/ft hr)	0.0483	1.4999			
Skin Visc (lbm/ft-hr)	0.0000	1.4777	Surface Effectiveness (Eta)	0.9745	
Density (lbm/ft")	0.0624	61.8749	Sensible Heat Transferred (BTU/hr) 10	07,615	
Cp (BŤU/lbm °F)	0.2402	0.9988	Latent Heat Transferred (BTU/hr)		
K (BTU/hr·ft·°F)	0.0160	0.3662	Heat to Condensate (BTU/hr)		
Relative Humidity In (%)	71.82				
Relative Humidity Out (%)	83.16				

** Reynolds Number Outside Range of Equation Appheability

*** Air Mass Velocity (Lbm hr ft^o), Tube Fluid Velocity (ft'sec); Air Density at Inlet T, Other Properties at Average T

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TVA

Calculation Report for BFN2CCL-070-740 - Drywell Air Cooler 150F, 50°4RH, EPU RAI 6 INITIAL CONDITION

Extrapolation Calculation for Row 4(Dry)

	Air-Side	Tube-Side	
Mass Flow (Ibm/hr)	68,850.34	63,353.58	Tube-Side hi (BTU/hr ft ^{2.} °F) 1,746.40
Inlet Temperature (°F)	130.20	106.65	j Factor 0.0102
Outlet Temperature (°F)	125.49	108.09	Air-Side ho (BTU/hr-ft [*] .°F) 15.31
Inlet Specific Humidity	0.084838		Tube Wall Resistance (hr-ft ² .ºF/BTU; 0.00016250
Outlet Specific Humidity	0.084838		Overall Fouling (hr ft ² °F/BTU) 0.00412058
Average Temp (°F)	127.84	107.3689	
Skin Temperature (°F)	109.79	108.6395	U Overall (BTU/hr·ft ^z ·°F) 13.15
Velocity ***	5.047.79	6.2565	Effective Area (ft [*]) 341.11
Reynold's Number	1.893++		LMTD 20.38
Prandti Number	0.7268	4.1643	Total Heat Transferred (BTU/hr) 91,416
Bulk Visc (lbm/ft-hr)	0.0480	1.5241	
Skin Visc (Ibm/ft-hr)	0.0000	1.5045	Surface Effectiveness (Eta) 0.9746
Density (lbm/ft')	0.0629	61.8968	Sensible Heat Transferred (BTU/hr) 91,416
Cp (BTU/ibm·°F)	0.2402	0.9989	Latent Heat Transferred (BTU/hr)
K (BTU/hr-ft-°F)	0.0159	0.3656	Heat to Condensate (BTU/hr)
Relative Humidity In (%)	83.16		
Relative Humidity Out (%)	94.42		

** Reynolds Number Outside Range of Equation Applicability

Extrapolation Calculation for Row 5(Dry)					
	Air-Side	Tube-Side			
Mass Flow (Ibm/hr)	68,850.34	63,353.58	Tube-Side hi (BTU/hr ft².°F)	1,736.14	
Inlet Temperature (°F)	125. 49	106.00	j Factor	0.0102	
Outlet Temperature (°F)	123.39	106.65	Air-Side ho (BTU/hr-ft ^z .°F)	15.28	
Inlet Specific Humidity	0.084838		Tube Wall Resistance (hr ft ² .ºF/BTU)	0.00016250	
Outlet Specific Humidity	0.084838		Overall Fouling (hr ft* °F/BTU)	0.00412058	
Average Temp (°F)	124.44	106.3245			
Skin Temperature (°F)	108.47	107.4529	U Overall (BTU/hr·ft*·°F)	13.12	
Velocity ***	5.047.79	6.2550	Effective Area (ft [*])	171.64	
Reynold's Number	1.902++	39,743	LMTD	18.09	
Prandtl Number	0.7271	4.2137	Total Heat Transferred (BTU/hr)	40,753	
Bulk Visc (lbm/ft-hr)	0.0478	1.5405			
Skin Visc (Ibm/ft-hr)	0.0000	1.5228	Surface Effectiveness (Eta)	0.9746	
Density (lbm/ft*)	0.0629	61.9111	Sensible Heat Transferred (BTU/hr)	40,753	
Cp (BTU/lbm·°F)	0.2402	0.9989	Latent Heat Transferred (BTU/hr)		
K (BTU/br·ft·°F)	0.0158	0.3652	Heat to Condensate (BTU/hr)		
Relative Humidity In (%)	94.42				
Relative Humidity Out (%)	100.00				

** Reynolds Number Outside Range of Equation Applicability

*** Air Mass Velocity (Lbm hr:fr), Tube Fluid Velocity (ft/sec); Air Density at Inlet T, Other Properties at Average T

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TVA

Calculation Report for BFN2CCL-070-740 - Drywell Air Cooler

150F, 50%RH, EPU RAI 6 INITIAL CONDITION

Extrapolation Calculation for Row 5(Wet)

	Air-Side	Tube-Side	
Mass Flow (lbm/hr)	68,850.34	63,353.58	Tube-Side hi (BTU/hr-ft ⁻ °F) 1,730.22
Inlet Temperature (°F)	123.39	104.12	Factor 0.0102
Outlet Temperature (°F)	122.82	106.00	Air-Side ho (BTU/hr-ft* °F) 75.22
Inlet Specific Humidity	0.084838		Tube Wall Resistance (hr-ft ² .ºF/BTU; 0.00016250
Outlet Specific Humidity	0.083331		Overall Fouling (hr·fl ² .ºF/BTU) 0.00412058
Average Temp (°F)	123.10	105.0625	
Skin Temperature (°F)	111.87	108.6458	U Overall (BTU/hr·ft [*] ·°F) 41.70
Velocity ***	5.047.79	6.2533	Effective Area (ft [*]) 169.47
Reynold's Number	1,905++	39,228	LMTD 18.02
Prandtl Number	0.7272	4.2746	Total Heat Transferred (BTU/hr) 127,362
Bulk Visc (lbm/ft-hr)	0.0477	1.5608	
Skin Visc (Ibm/ft-hr)	0.0000	1.5044	Surface Effectiveness (Eta) 0.8902
Density (lbm/ft')	0.0633	61.9282	Sensible Heat Transferred (BTU/hr) 10,997
Cp (BTU/lbm·°F)	0.2402	0.9989	Latent Heat Transferred (BTU/hr) 116,371
K (BTU/hr·ft·°F)	0.0158	0.3647	Heat to Condensate (BTU/hr) 8,589
Relative Humidity In (%)	100.00		
Relative Humidity Out (%)	100.00		

** Reynolds Number Outside Range of Equation Applicability

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Extrapolation Calculation for Row 6(Wet)					
	Air-Side	Tube-Side			
Mass Flow (lbm/hr)	68,850.34	63,353.58	Tube-Side hi (BTU/hr·ft²·°F)	1,702.67	
Inlet Temperature (°F)	122.82	100.01	j Factor	0.0102	
Outlet Temperature (°F)	121.53	104.12	Air-Side ho (BTU/hr ft*.°F)	72.77	
Inlet Specific Humidity	0.083331		Tube Wall Resistance (hr ft ² .ºF/BTU)	0.00016250	
Outlet Specific Humidity	0.080042		Overall Fouling (hr-ft ² . °F/BTU)	0.00412058	
Average Temp (°F)	122.17	102.0681			
Skin Temperature (°F)	109.55	106.0385	U Overall (BTU/hr·ft*·°F)	40.80	
Velocity ***	5,047.79	6.2493	Effective Area (ft [*])	341.11	
Reynold's Number	1,907++	38,015	LMTD	20.03	
Prandtl Number	0.7273	4.4250	Total Heat Transferred (BTU/hr)	278,758	
Bulk Visc (lbm/ft-hr)	0.0476	1.6106			
Skin Visc (Ibm/ft·hr)	0.0000	1.5451	Surface Effectiveness (Eta)	0.8932	
Density (lbm/ft*)	0.0638	61.9678	Sensible Heat Transferred (BTU/hr)	24,852	
Cp (BTU/lbm·°F)	0.2402	0.9989	Latent Heat Transferred (BTU/hr)	25 3,89 3	
K (BTU/hr·ft·°F)	0.0157	0.3636	Heat to Condensate (BTU/hr)	18,300	
Relative Humidity In (%)	100.00				
Relative Humidity Out (%)	100.00				

** Reynolds Number Outside Range of Equation Applicability

*** Air Mass Velocity (Lbm, hr.fr), Tube Fluid Velocity (ft/sec); Air Density at Inlet T, Other Properties at Average T

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