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TN Calculation 1121-0401, Revision **1,**

0S197L 75 Ton Transfer Cask Thermal Analysis to be used with OPPD Exemption Request (18.4 **kWIDSC & 11.0 kWIDSC**

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1.0 PURPOSE

The purpose of the calculation is determine 32PT **DSC** basket components and fuel cladding temperatures within the **0S1 97L (75** ton) cask with 18.4 **kW/DSC** and **11.0 kW/DSC** total heat load for off-normal transfer condition. This is the controlling case (lowest margin to fuel cladding temperature limit) during fuel load and transfer operation. The body of the calculation addresses the 16.4 kW heat load case and Appendix **A** addresses the **11. 0** kW condition.

2.0 ASSUMPTIONS/CONSERVATISMS

The assumptions and conservatism described in NUHOMS[®] 32PT DSC thermal evaluation **[11** are applied in this calculation.

3.0 DESIGN INPUT - 18.4 KW

The material properties listed in **[1]** are used in the analysis.

1) The fuel assembly used in **[11** is used as the limiting assembly type in this calculation based on the determination of limiting fuel effective conductivity among the fuel assemblies that are considered to be stored in 32PT **DSC.** Total heat load is 18.4 **kW/DSC.** The heat zone configuration is shown in Figure **3-1.**

Figure 3-1 Fuel Loading Configuration (18.4 kWIDSC)

***** This Is a very conservative assumption for the fuel assemblies **(CE** 14x14) to be loaded at OPPID In the 32PT **DSC.** Based on Reference **[1], CE** 14x14 fuel assembly has higher effective fuel conductivities compared to design basis fuel assembly used In **[1].** Moreover, the use of **0.7 I(W/FA** for fuel assemblies In all basket center locations Is also very conservative for OPPD fuel loading. It is expected that fuel assemblies to be loaded at OPPD site will have significantly lower decay heat than **0.7** kW.

2) Summary of ambient conditions considered in the analysis is shown in Table **3-1** below.

~07 1 0F shall be used as the average ambient air temperature for the steady state maximum off-normal condition [1].

4.0 METHODOLOGY

The methodology used for thermal analysis is the same as described in **[3].** The **DSC** shell temperature profile is calculated *using a* two dimensional thermal model of the OS197L cask and DSC shell, and uses the ANSYS [2] computer program. These DSC shell temperatures are than used as a constant temperature boundary condition in an **ANSYS** model of the 32PT **DSC** and basket to calculate the fuel cladding temperature **[1].** These fuel cladding temperatures are then used to calculate the effect of the transfer cask skid shielding as described in Section **5.3.**

5.0 FINITE ELEMENT MODELS

5.1 2D 0S197L Cask Thermal Model

The **DSC** shell within the **0S197L** cask is analyzed for the operating conditions listed in Table **3-1** for 18.4 **kW/DSC** heat load.

The **ANSYS** models of the **0S197L (75** ton) transfer cask including the **DSC** shell represent a two-dimensional slice of the **OS 197L** cask at the axial centerline as shown in Figure **5-1** through Figure **5-2.**

Figure 5-1 OS197L (75 ton) Transfer Cask ANSYS Model

Figure 5-2 Details of 0S197L (75 ton) Transfer Cask ANSYS Model

The material properties and cask dimensions are taken from **[3].** Only the total heat load per **DSC** is changed to 18.4 kW from 24 kW

The method of the heat flux calculation is consistent with the **OS 197L** analysis methodology **[3].** Note that any heat removed **by** the ends of **DSC** is conservatively neglected in calculation of heat flux. The heat flux is calculated based on the 32PT **DSC** shell length **[1]** as:

$$
\ddot{q} = \frac{18.4 \text{kW} \cdot 3412.3 \frac{\text{Btu}}{\text{kW}}}{60 \frac{\text{min}}{\text{hr}} \cdot (\pi \cdot 66 \text{in} \cdot 182 \text{in})} = 0.0277 \frac{\text{Btu}}{\text{min} \cdot \text{in}^2}.
$$

The radiation is modeled **by** overlaying surface elements and using the **/AUXI2** processor to compute view factors. Radiation to the environment is modeled with **SURF1 51** elements in **ANSYS.** Radiation between the **OSO** shell and cask structural shell, between the cask structural shell and the cask inner neutron shield panel (or steel shielding) is also modeled using the **/AUX12** processor in ANSYS [3].

The convection to the ambient is conservatively based on average film temperature for convection coefficient evaluation in the **ANSYS** model. The convection coefficients are calculated based on correlation of turbulent natural convection for horizontal cylinder [4]:

$$
h = 0.18(\Delta T)^{1/3} \frac{Btu}{hr - ft^2 \text{ °F}} = 2.083e - 5(\Delta T)^{1/3} \frac{Btu}{min - inch^2 \text{ °F}} \text{ for } 10^9 < Gr_L < 10^{12}.
$$

The heat is applied to the model as a heat flux on the inner surface of the **DSC** using **SURFIS51** elements in **ANSYS.**

5.2 3D 32PT **DSC** Thermal Model

The **DSC** outer shell temperatures are based on a **2D** DSC/cask model, which assumes no heat transfer in the axial direction. This **2D 0S1 97L** model produces conservative (higher) **DSC** shell temperatures, which are used as input to the **3D DSC** thermal model, which calculated the fuel clad temperatures. Both the normal (with insolation) and off-normal **0S197L** transfer conditions with 18.4 **kW/DSC** are calculated.

The same basket component and fuel properties from **[1]** are used. Heat generations are calculated based on the dimensions of the fuel and basket. The heat is assumed to be distributed evenly through the **8.7"** square nominal fuel cell opening. Axial variations are accounted for in **ANSYS by** using the peaking factors similar to **[1].** The base heat generations with the corresponding peaking factors are applied according to the loading patterns given in Figure **3-1:**

For **0.7** kW heat load:

$$
\ddot{q} = \frac{0.7kW \cdot 3414 \frac{Btu}{kW} \cdot \frac{1hr}{60 \text{ min}}} {(8.7in)^2 \cdot 141.8in} = 3.711e - 3 \frac{Btu}{min \cdot in^3}.
$$

For **0.5** kW heat load:

$$
\ddot{q} = \frac{0.5kW \cdot 3414 \frac{Btu}{kW} \cdot \frac{1hr}{60 \text{ min}}}{(8.7in)^{2} \cdot 141.8in} = 2.65e - 3 \frac{Btu}{\text{ min} \cdot in^{3}}.
$$

5.3 Methodology to Determine Maximum Fuel Cladding Temperatures in **0S197L** Transfer Cask with Skid Shielding

An additional skid shielding prevents insolation on the cask surface but effects the convective flow at the outer cask surface. The **3D 051 97L** caskIDSC shell **CFD** model was developed to analyze this effect on the cask component temperatures **[5].**

The effect of the skid shielding on the maximum fuel cladding temperature was evaluated **by** the following steps:

- **1)** The average temperature of the outer surface of neutron shield outer panel T_{out NS} **P** aver was calculated **by CFD** run **[5]** and used to obtain the average **DSC** shell temperature **Tsh av by** extrapolation of the results **Tsh av =** f(Tout **NS paver)** available from the **2D ANSYS** runs for hot off-normal (117°F ambient, no insolation) and hot normal (100°F ambient insolation) cases calculated in Section **5. 1.**
- 2) The average DSC shell temperature T_{sh av}, calculated above, was then used to obtain the maximum fuel cladding temperature T_{fuel} max by extrapolation of the results $T_{fuel max}$ = **f(Tsh av)** available from the **DSC** within **081 97L-1** runs in Section **5.2** for hot off-normal (117°F ambient, no insolation) and hot normal (100°F ambient, insolation) for the 08 **1 97L** cask with skid shielding (See Section 5.2 above for **Tfuel max** calculation).

6.0 REFERENCES

- 1. Calculation, NUHOMS[®] 32PT DSC Thermal Evaluation for 10CFR, Part 72 Storage Conditions, Transnuclear, Inc. Calculation No. **NUH** 32PT.0403, Rev **3.**
- 2. On-Line User's Manual for **ANSYS,** Revision **8. 1.**
- **3.** Calculation, Thermal Analysis of **0S197L** and **0S197L 100** Transfer Cask, Transnuclear, Inc. Calculation No. **NUHO6L-0400,** Rev 2.
- 4. Ozisik, **N.** M., Basic Heat Transfer, McGraw Hill Book Company, **1977.**
- **5.** Calculation, Calculation of **0S197L** Cask Shell Temperature with 18.4 kW and **11. 0** kWHeat Load, Transnuclear, Inc. Calculation No. 1121.0400. Rev. **1.**
- **6.** Calculation, Minimum Fuel Effective Conductivity for 32PT Design, Transnuclear, Inc. Calculation No. 60220-14, Rev **0.**

7.0 COMPUTATIONS

The **ANSYS 8.1** [2] model was runs on Xeon **3.2** GHz platform. The runs are summarized in Table **7-1** below. The input, output **ANSYS** database, and result files are located on the **CD,** which accompany this calculation.

8.0 RESULTS AND SUMMARY

8.1 Temperature of **0S197L** Components, and 18.4 **kW DSC** Shell

The temperature plots for the **DSC** shell and the **0S197L** transfer cask for normal operating conditions (with insolation) and off-normal (Without insolation) are shown in Figure **8-1** and Figure **8-2,** respectively.

I

Summary of the 18.4 kW **DSC** shell, and maximum **0S197L** transfer cask component temperatures are shown in Table **8-1** below.

Table **8-1 DSC** Shell and **0S1971** Cask Component Maximum Temperatures, **ANSYS** Results (18.4 **kWIDSC)**

(1) Conservative value is applied in **DSO** thermal model.

(2) Extrapolated value based on **2D** cask **ANSYS** runs for off-normal and normal transfer **OS197L** cases

where

T_{top}, T_{side}, T_{bot} – DSC shell top, side and bottom maximum temperature, respectively,

Tsh **av - DSC** shell inner surface average temperature,

Tout **ISS 1) -** cask outer **NS** panel average temperature.

8.2 Evaluation of Maximum Fuel Cladding Temperature during Transfer in OS197L

The **DSC** shell top, side, and bottom temperatures are used in this calculation. An average DSC shell temperature T_{sh av} was calculated as the average nodal temperature at the inner **DSC** shell diameter. This average temperature is then used to calculate maximum fuel cladding temperature for 18.4 **kW/DSC** transferred within the **0S197L.** The results are shown in Table **8-2** below.

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Table 8-2 **DSC** Shell and Fuel Cladding Maximum Temperatures for **0S197L** Cask **ANSYS** Results (18.4 **kWIDSC)**

(1) See Table **8-1.**

(2) Extrapolated value.

6.3 Effect of Skid Shielding on Thermal Analysis Results

As seen from Table **8-2,** the maximum fuel cladding temperature for 18.4 kW /32PT **DSC** within **0S197L** are below allowable limits for normal and off-normal transfer conditions without skid shielding and for off-normal transfer conditions with skid shielding. The normal condition with skid shielding is bounded **by** the off-normal as the skid shielding eliminates the insolation and the ambient temperature is higher.

A.1 **Purpose**

The purpose of this Appendix **A** is to determine 32PT **DSC** basket components and fuel cladding temperatures within the *OS* **197L (75** ton) cask with **11. 0 kW/DSC** total heat load for off-normal transfer condition.

A.2 Assumptions/Conservatisms

There is no change to the assumptions or conservatisms, and those of **[1]** are applied in this calculation.

A.3 Design Input- 11.0kW

This Appendix **A** provides a thermal evaluation of the maximum fuel cladding temperature for an **11. 0 kW/DSC** within **OS 197L.**

The thermal evaluation is similar to the evaluation described previously in Sections 2.0 through **5.0** for the 18.4 **kW/DSC** total heat load.

Three different design inputs are used in this Appendix **A:**

A. 3. I The heat flux applied on **DSC** shell inner surface for **11 kW/DSC** is:

$$
\ddot{q} = \frac{11kW \cdot 3412.3 \frac{Btu}{kW}}{60 \frac{min}{hr} \cdot (\pi \cdot 66 in \cdot 182 in)} = 0.0166 \frac{Btu}{min \cdot in^2}.
$$

A.3.2 The effective fuel thermal conductivity of the fuel assembly **CE** 14x14-FC is listed in Appendix Table **A-I [61,** and was used in the model.

- **A.3.3** The fuel zoning configuration was redefined to reduce peak fuel clad temperatures and to incorporate the lower **11.0 kW/DSC** heat loading. The configuration **is shown** in Appendix Figure **A-I.**
- A.3.4 The base heat generation with the corresponding axial peaking factor is applied according to the loading pattern given in the fuel loading configuration for *I11* **kW/DSC,** as shown in Appendix Figure **A-I.**

 $\label{eq:2.1} \frac{1}{\sqrt{2}}\int_{\mathbb{R}^3} \frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\left(\frac{1}{\sqrt{2}}\right)^2\left(\frac{1}{\sqrt{2}}\right)^2\left(\frac{1}{\sqrt{2}}\right)^2\left(\frac{1}{\sqrt{2}}\right)^2.$

Appendix Figure A-I Fuel Loading Configuration **(11 kWIDSC)**

'This is **a** bounding fuel toad for the fuel assemblies **(CE** 14x14) to be loaded at OPPID in the 32PT **DSC.** Conservatively, the total heat generation used In the **3D DSC** thermal model based on the above fuel loading configuration is 12.64 kW/ **DSC.** The total heat load to be loaded at OPPD site at the Phase **I** campaign will be less than **11** kW per **DSC.**

 $\begin{split} \mathcal{L}_{\text{max}} & \mathcal{L}_{\text{max}} \\ & \mathcal{L}_{\text{max}} & \mathcal{L}_{\text{max}} \\ & \mathcal{L}_{\text{max}} & \mathcal{L}_{\text{max}} \\ & \mathcal{L}_{\text{max}} & \mathcal{L}_{\text{max}} \end{split}$

A.4 Methodology

There is no change to the methodology used to calculate the peak fuel clad temperatures, described previously in Section 4.0.

A. 5 Finite Element Models

The same models, as were used previously for the 18.4 **kW/DSC** calculation, were used in this 11.0 kW/DSC calculation. The differences were the reduction in heat flux detailed above and the reduction in base fuel assembly heat generations within the basket, which correspond to the reduced 0.40, **0.35,** and **0. 16** kW/Assembly Zones (Zones **I** through **3).**

A.6 References

There is no change to the references.

A. 7 Computations

All computer runs for the **11. 0** kW conditions are listed in Section **7. 0.**

A. 8 Results and Summar

The **ANSYS** runs for **11 kW/DSC** heat load are listed in Table **7-1.**

The temperature plots for the **DSC** shell and **0S197L** transfer cask for normal operating conditions (with insolation) and off-normal (without insolation) for **11 kW/DSC** heat load are shown in Appendix Figure **A-2** and Appendix Figure **A-3,** respectively.

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The summary of the **DSC** shell and **OS 197L** transfer cask component temperatures is shown in Appendix Table **A-2** below.

Appendix Figure A-4 and Appendix Figure **A-5** show **DSC** component and fuel temperature plots for off-normal and normal transfer conditions for the **11 kW/DSC** heat load.

Appendix Figure A-4 Temperature Plot for Transfer of **11 kWIDSC within 0S1 97L** for Off-Normal Conditions **(117 0F,** *No* Solar), No **Skid** Shielding

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Appendix Table **A-2 DSC** Shell Temperatures, Average Outer Neutron Shield Panel **Temperatures and Maximum Fuel Cladding Temperatures (11 kW/DSC)**

(1) Conservative value is applied in **DSC** thermal model.

(2) Extrapolated value based on **2D** cask **A NSYS** runs for off-normal and normal transfer **0S197L** cases

(3) Extrapolated value based on **3D DSC ANSYS** runs for off-normal and normal transfer **0S197L** cases

(4) Calculated using **CFD** model **[5)**

As seen from Appendix Table **A-2,** the maximum fuel cladding temperature for **11** kW 132PT **DSC** within **OS1 97L** are below allowable limits for normal and off-normal transfer conditions without skid shielding, and for off-normal conditions with skid shielding. The normal condition with skid shielding is bounded **by** the off-normal as the skid shielding eliminates the insolation and the ambient temperature is higher.