

Westinghouse Non-Proprietary Class 3

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Indian Point Nuclear Generating Unit No. 2

Stretch Power Uprate NSSS and BOP Licensing Report

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LIST OF ACRONYMS

| | |
|----------|--|
| 1-D | one-dimensional |
| 2-D | two-dimensional |
| 3-D | three-dimensional |
| AAC | alternate AC |
| AEC | Atomic Energy Commission |
| AFW | auxiliary feedwater |
| AFWS | Auxiliary Feedwater System |
| AISC | American Institute of Steel Construction |
| ALARA | as-low-as-is-reasonably-achievable |
| AMSAC | ATWS mitigating system actuation circuitry |
| ANC | Advanced Nodal Code |
| ANS | American Nuclear Society |
| ANSI | American National Standards Institute |
| AOR | Analysis of Record |
| ART | adjusted reference temperature |
| ARV | atmospheric relief valve |
| ASD | after shutdown |
| ASME | American Society of Mechanical Engineers |
| ASSS | Alternate Safe Shutdown System |
| AST | alternative source term |
| ATWS | anticipated transient without scram |
| AV | allowable value |
| AVB | anti-vibration bar |
| B&PV | boiler and pressure vessel |
| BELBLOCA | best-estimate large-break loss-of-coolant accident |
| BFRV | bypass feedwater regulator valve |
| bhp | brake horsepower |
| BMI | bottom-mounted instrumentation |
| BOC | beginning of cycle |

LIST OF ACRONYMS (Cont.)

| | |
|---------|---------------------------------------|
| BOL | beginning of life |
| BOP | balance of plant |
| BOT | break opening time |
| Btu | British thermal unit |
| C&FS | Condensate and Feedwater System |
| CAOC | constant axial offset control |
| CCR | central control room |
| CCW | component cooling water |
| CCWS | Component Cooling Water System |
| CDF | core damage frequency |
| CEDE | committed effective dose equivalent |
| CFD | computational fluid dynamics |
| CFR | Code of Federal Regulations |
| CLOF | complete loss of flow |
| CLOF-UF | complete loss of flow under frequency |
| CN | calculation note |
| COLR | Core Operating Limit Report |
| COMS | Cold Overpressure Mitigation System |
| CR | containment recirculation |
| CRDM | control rod drive mechanism |
| CS | containment spray |
| CSA | channel statistical allowance |
| CSS | Containment Spray System |
| CST | condensate storage tank |
| CUF | cumulative usage factor |
| Cv | valve flow coefficient |
| CVCS | Chemical and Volume Control System |
| CW | circulating water |
| CWIT | circulating water inlet temperature |

LIST OF ACRONYMS (Cont.)

| | |
|------|---------------------------------------|
| CWS | Circulating Water System |
| DBA | design basis accident |
| DBE | design basis earthquake |
| DCF | dose conversion factor |
| DCP | design change package |
| DDE | deep dose equivalent |
| DE | dose equivalent |
| DECL | double-ended cold leg |
| DEHL | double-ended hot leg |
| DEPS | double-ended pump suction |
| DER | double-ended rupture |
| DF | decontamination factor |
| DG | diesel generator |
| DNB | departure from nucleate boiling |
| DNBR | departure from nucleate boiling ratio |
| DOR | Division of Operating Reactors |
| dpa | displacement of atom |
| DSS | Diverse Scram System |
| DW | Direct Work Items |
| EAB | exclusion area boundary |
| EBOP | emergency bearing oil pump |
| ECCS | Emergency Core Cooling System |
| EDE | effective dose equivalent |
| EDG | emergency diesel generator |
| EFPY | effective full-power year |
| EM | evaluation model |
| EOC | end of cycle |
| EOL | end of life |
| EOP | Emergency Operating Procedure |

LIST OF ACRONYMS (Cont.)

| | |
|-------|--|
| EPA | Environmental Protection Agency |
| EPRI | Electric Power Research Institute |
| EPU | extended power uprate |
| EQ | environmental qualification |
| ERG | Emergency Response Guideline |
| ES | Extraction Steam |
| ESF | engineered safety feature |
| ESFAS | Engineered Safety Feature Actuation System |
| ESOP | emergency seal oil pump |
| ET | Electric Tunnel |
| FAC | final acceptance criteria |
| FAC | flow-accelerated corrosion |
| FACP | Flow Accelerated Corrosion Program |
| FCEP | Fuel Criteria Evaluation Process |
| FCU | fan cooling unit |
| FCV | feedwater control valve |
| FDB | flow distribution baffle |
| FES | Final Evaluation Statement |
| FHA | fuel-handling accident |
| FHB | Fuel-Handling Building |
| FIV | feedwater isolation valve |
| FIV | flow-induced vibration |
| FLB | feedwater line break |
| F_N | Froude Number |
| FOA | fans, oil, and air |
| FPPP | Fire Protection Program Plan |
| FQ | peaking factor |
| FRV | feedwater regulator valve |
| FSAR | Final Safety Analysis Report |

LIST OF ACRONYMS (Cont.)

| | |
|-------|--|
| FU | fuel upgrade |
| FWH | feedwater heater |
| FWI | feedwater isolation |
| FWIV | feedwater isolation valve |
| FWS | Feedwater System |
| GDC | General Design Criteria |
| GDT | gas decay tank |
| GL | Generic Letter |
| GSI | Generic Safety Issue |
| GSS | Gland Steam System |
| GWDS | Gaseous Waste Disposal System |
| HEI | Heat Exchange Institute, Inc. |
| HELB | high-energy line break |
| HFF | hydraulic forcing function |
| HFP | hot full power |
| HHSI | high-head safety injection |
| HHSIS | High-Head Safety Injection System |
| HLSO | hot leg switchover |
| hp | horsepower |
| HP | high pressure |
| HT | holdup tank |
| HVAC | heating, ventilation, and air conditioning |
| HZP | hot zero power |
| I&C | instrumentation and control |
| ICH | in-core hold time |
| ID | inside diameter |
| IFBA | integral fuel burnable absorber |
| IFM | intermediate flow mixing |
| IGSCC | intergranular stress corrosion cracking |

LIST OF ACRONYMS (Cont.)

| | |
|-----------|---|
| IP1 | Indian Point Unit 1 |
| IP2 | Indian Point Unit 2 |
| IP3 | Indian Point Unit 3 |
| IPB | Iso-Phase bus |
| ISI | in-service inspection |
| ISLH | in-service leak and hydrostatic |
| IST | in-service testing |
| ITS | Improved Technical Specifications |
| K_{eff} | effective multiplication factor |
| K_I | stress intensity factor |
| K_{IC} | critical value of K_I , or fracture toughness |
| K_{IR} | reference stress intensity factor |
| LAR | Licensing Amendment Request |
| LBB | leak before break |
| LBLOCA | large-break loss-of-coolant accident |
| LCV | level control valve |
| LEFM | linear elastic fracture mechanics |
| LERF | large early release frequency |
| LHF | LOCA hydraulic force |
| LHSI | low-head safety injection |
| LHSIS | Low-Head Safety Injection System |
| LOAC | loss-of-AC power |
| LOCA | loss-of-coolant accident |
| LONF | loss of normal feedwater |
| LOOP | loss-of-offsite power |
| LP | low pressure |
| LPP | low pressurizer pressure |
| LPZ | low-population zone |
| LTOP | low-pressure overpressure protection |

LIST OF ACRONYMS (Cont.)

| | |
|--------|--|
| LTOPS | Low-Pressure Overpressure Protection System |
| LWR | light water reactor |
| M&E | mass and energy |
| MA | mill-annealed |
| MBFP | main boiler feed pump |
| m/c | measurement/calculation |
| MCO | moisture carryover |
| MDAFWP | motor-driven auxiliary feedwater pump |
| MFIV | main feedwater isolation valve |
| MFP | main feedwater pump |
| MFV | main feedwater valve |
| MMF | minimum measured flow |
| MOC | middle of cycle |
| MOL | middle of life |
| MOV | motor-operated valve |
| MS | main steam |
| MSIV | main steam isolation valve |
| MSLB | main steamline break |
| MSR | moisture separator reheater |
| MSS | Main Steam System |
| MSSV | main steam safety valve |
| MT | main transformer |
| MTC | moderator temperature coefficient |
| MTU | metric ton unit |
| MUR | measurement uncertainty recapture |
| NDE | nondestructive examination |
| NEC | National Electric Code |
| NEMA | National Electric Manufacturer's Association |
| NIS | Nuclear Instrumentation System |

LIST OF ACRONYMS (Cont.)

| | |
|---------------|--|
| NPSH | net positive suction head |
| NPSHA | net positive suction head, actual |
| NPSHR | net positive suction head, required |
| NR | narrow range |
| NRC | Nuclear Regulatory Commission |
| NRS | narrow range span |
| NSSS | Nuclear Steam Supply System |
| NTS | nominal trip setpoint |
| NUMARC | Nuclear Management and Resource Council |
| NUPPSCO | Nuclear Power Plant Standards Committee |
| NYISO | New York Independent System Operator |
| OBE | operating basis earthquake |
| OD | outside diameter |
| ODSCC | outer diameter stress corrosion cracking |
| OEM | Original Equipment Manufacturer |
| OFA | optimized fuel assembly |
| OL | Operating License |
| OPS | Overpressure Protection System |
| OP Δ T | overpower Δ T |
| OT Δ T | overtemperature Δ T |
| P&I | proportional and integral |
| PAB | Primary Auxiliary Building |
| PAOT | post accident operability time |
| PCT | peak clad temperature |
| PCWG | Performance Capability Working Group |
| PLOF | partial loss of flow |
| PICS | Plant Integrated Computer System |
| PORV | power-operated relief valve |
| POV | power-operated valve |

LIST OF ACRONYMS (Cont.)

| | |
|-------|--|
| PRT | pressurizer relief tank |
| PSA | Probabilistic Safety Assessment |
| PSS | Primary Sampling System |
| PSV | pressurizer safety valve |
| P-T | pressure-temperature |
| PTS | pressurized thermal shock |
| PU | power uprate |
| PVC | polyvinyl chloride |
| PWR | pressurized-water reactor |
| PWSCC | primary water stress corrosion cracking |
| PWST | primary water storage tank |
| PZR | pressurizer |
| QA | Quality Assurance |
| RAI | Request for Additional Information |
| RAT | reserve auxiliary transformer |
| RCCA | rod control cluster assembly |
| RCFC | reactor containment fan cooler |
| RCL | reactor coolant loop |
| RCP | reactor coolant pump |
| RCS | Reactor Coolant System |
| RCSES | Reactor Coolant System equipment support |
| RG | Regulatory Guide |
| RHR | residual heat removal |
| RHRS | Residual Heat Removal System |
| RI | reactor internals |
| RPS | Reactor Protection System |
| RPV | reactor pressure vessel |
| RSAC | reload safety analysis checklist |
| RSE | Reload Safety Evaluation |

LIST OF ACRONYMS (Cont.)

| | |
|-------------------|---|
| RSG | replacement steam generator |
| RTD | resistance temperature detector |
| RTDP | Revised Thermal Design Procedure |
| RT _{NDT} | reference temperature nil ductility temperature |
| RTP | rated thermal power |
| RT _{PTS} | reference temperature-pressurized thermal shock |
| RTS | Reactor Trip System |
| RV | reactor vessel |
| RVHP | reactor vessel head penetration |
| RWST | refueling water storage tank |
| S&W | Stone and Webster |
| SAL | safety analysis limit |
| SAT | station auxiliary transformer |
| SB | site boundary |
| SBLOCA | small-break loss-of-coolant accident |
| SBO | station blackout |
| SBV | Shield Building ventilation |
| SCC | stress corrosion cracking |
| SER | <i>Safety Evaluation Report</i> |
| SFP | spent fuel pit |
| SFPCS | Spent Fuel Pit Cooling System |
| SG | steam generator |
| SGBS | Steam Generator Blowdown System |
| SGR | steam generator replacement |
| SGTP | steam generator tube plugging |
| SGTR | steam generator tube rupture |
| SI | safety injection |
| SIS | Safety Injection System |
| SJAE | steam jet air ejector |

LIST OF ACRONYMS (Cont.)

| | |
|------------|--|
| SLI | steamline isolation |
| SP | separator parameter |
| SPDES | State Pollutant Discharge Elimination System |
| SPU | stretch power uprate |
| SRIS | System Reliability Impact Study |
| SRP | Standard Review Plan |
| SRSS | square root sum of the squares |
| SSE | safe shutdown earthquake |
| STDP | Standard Thermal Design Procedure |
| SW | service water |
| SWGR | switchgear room |
| SWPC | Siemens-Westinghouse Power Corporation |
| SWS | Service Water System |
| TA | total allowance |
| T_{avg} | average temperature ¹ |
| T_{cold} | cold leg temperature |
| TDAFWP | turbine-driven auxiliary feedwater pump |
| TDF | thermal design flow |
| TDH | total discharge head |
| TEDE | total effective dose equivalent |
| TGSCC | transgranular stress corrosion cracking |
| T_{hot} | hot leg temperature |
| TID | Technical Information Document |
| t_{min} | tube wall thickness minimum |
| t_{nom} | tube wall thickness nominal |
| TOI | Temporary Operation Instruction |
| T_{ref} | reference temperature ¹ |

¹ RCS or vessel temperature, depending on context.

LIST OF ACRONYMS (Cont.)

| | |
|--------------------|--|
| T_{sat} | water at pressurizer temperature or saturation temperature |
| TSP | trisodium phosphate |
| TSP | tube support plate |
| T_{steam} | steam temperature |
| UAT | unit auxiliary transformer |
| UFSAR | <i>Updated Final Safety Analysis Report</i> |
| UHS | ultimate heat sink |
| USE | upper shelf energy |
| UT | ultrasonic testing |
| WBS | Work Breakdown Structure |
| VCT | volume control tank |
| WCAP | Westinghouse Commercial Atomic Power |

1.0 INTRODUCTION

1.1 Background

This report supports the United States Nuclear Regulatory Commission (NRC) review and approval of the Indian Point Unit 2 (IP2) Stretch Power Uprate (SPU) License Amendment Request (LAR). IP2 is presently licensed for a core power rating of 3114.4 MWt. The SPU will increase the IP2 licensed core thermal power to 3216 MWt.

This report summarizes the various analyses and evaluations of the potential effects of the SPU on plant systems, components, and analyses.

1.1.1 Uprate Power Level

IP2 was originally licensed to operate with a rated core thermal power of 2758 MWt. By letter dated September 30, 1988, Con Edison initiated a request to authorize an increase in the licensed core thermal power level to 3071.4 MWt. The NRC documented their acceptance of this request in a *Safety Evaluation Report (SER)* dated March 7, 1990 (Reference 1). The current IP2 operating license issued by the NRC is for a rated reactor core power of 3114.4 MWt, based on the recently approved 1.4-percent measurement uncertainty recapture (MUR) uprate (Reference 2).

However, when originally licensed, various IP2 systems and components (engineered safety features in particular) were designed to accommodate the conditions (higher flows and temperatures) associated with a rated core thermal power of 3216 MWt, which is above the original licensed core thermal power (2758 MWt) and above the current licensed core thermal power (3114.4 MWt).

Also, since the original licensing, Westinghouse has maintained responsibility for the *Updated Final Safety Analysis Report (UFSAR)* (Reference 3), Chapter 14, safety analyses. Over the years since the original licensing, for some Chapter 14 analyses (large-break loss-of-coolant accident [LBLOCA] and radiological dose), Westinghouse has been able to maintain the use of the original licensed core thermal power of 3216 MWt. Therefore, both the engineering safeguards equipment design and some current plant safety analyses already support a higher core power (3216 MWt) than the core power that is currently licensed.

The Atomic Energy Commission (AEC) SER, Supplement 2, issued July 1971 (Reference 4) states, "Our evaluation of the engineered safety features (with the exception of the Emergency Core Cooling System) and our accident analyses, have been performed for a maximum power of 3216 MWt."

Continuing improvements in analytical techniques, instrument measurement accuracies, plant thermal performance, and fuel and core designs have resulted in increased margins between the safety analyses results and the licensing limits. These available safety margins, combined with the margins in the as-designed equipment, system, and component capabilities, and margins in the current safety analyses, provide IP2 with the opportunity to increase the current licensed thermal power rating of 3114.4 MWt to 3216 MWt (an increase of 3.26 percent) with no significant increase in the hazards presented by the plant as currently licensed by the NRC.

This was confirmed prior to full initiation of the SPU project when Entergy Nuclear Operations, Incorporated (Entergy) completed a feasibility and scoping study with the support of Westinghouse (Nuclear Steam Supply System), Stone & Webster (balance of plant), and the Siemens-Westinghouse Power Corporation (high-pressure turbine).

1.1.2 References

1. Letter from D. S. Brinkman (NRC) to S. B. Bram (Nuclear Power Consolidated Edison Company), *Issuance of Amendment No. 148, TAC No. 69542, SER for 1990 Stretch Power Upgrading*, March 7, 1990.
2. *Indian Point Nuclear Generating Unit No. 2 1.4-Percent Measurement Uncertainty Recapture Power Uprate License Amendment Request Package*, Entergy Nuclear Operations, Inc., November 2002.
3. *Indian Point Nuclear Generating Unit No. 2, Updated Final Safety Analysis Report*, Docket No. 50-247.
4. *Atomic Energy Commission (AEC) Safety Evaluation Report (SER)*, Supplement 2, July 1971.

1.2 Licensing Approach

1.2.1 Introduction

The NRC defines three categories of power uprates:

- Measurement uncertainty recapture (MUR) power uprates
- Stretch power uprates (SPUs)
- Extended power uprates (EPUs)

MUR power uprates are less than 2 percent. SPUs are typically up to 7 percent, and EPUs are greater than SPUs, and have been submitted to the Nuclear Regulatory Commission (NRC) for increases as high as 20 percent.

The Indian Point Unit 2 (IP2) SPU represents a licensed core power level increase of 3.26 percent. This level of uprate is more than what is typically considered for an MUR power uprate (NRC guidance in Regulatory Issue Summary [RIS] 2002-03, *Guidance on the Content of Measurement Uncertainty Recapture Power Uprate Applications*, January 31, 2002 [Reference 1]), but is significantly less than the 7-percent threshold defined by the NRC as the lower bound for EPU according to RS-001, (Reference 2) NRC guidance for review of EPUs. The NRC has not yet issued guidance pertaining to SPU programs. Therefore, this application incorporates appropriate elements of both the NRC MUR and EPU guidance documents.

While RIS 2002-03 (Reference 1) (MUR guidance) does not specifically apply to the 3.26-percent IP2 SPU, this report has been structured to clearly distinguish affected and unaffected plant systems, components, and analyses. Affected systems, components, and safety analyses are those having current design and licensing bases analyses and calculations that do not bound the potential effects of the SPU. Unaffected systems, components, and safety analyses are those having current design and licensing bases analyses and calculations that bound the potential effects of the SPU. This report also identifies whether affected plant systems, components, and analyses were addressed through analysis or engineering evaluation.

While RS-001 (Reference 2) (EPU guidance) does not explicitly apply to the 3.26-percent IP2 SPU, significant detail has been provided for the analyses and evaluations of affected systems, components, and analyses. In particular, more detail has been provided for the safety analyses since many of these analyses have been revised to address the increased power level, or revised to amend inputs and parameters to provide additional margin for operations. Also, this report is based upon the consideration of the EPU guidance regarding the scope of NRC's

review, and information expected in a power uprate application as discussed in the RS-001 (Reference 2).

The subject matter and detail of this report far exceeds that corresponding to the MUR guidance for power uprate. The full scope of this project was jointly established by Entergy, Westinghouse, Stone & Webster (S&W), and Siemens-Westinghouse Power Corporation (SWPC) as part of an extensive planning effort. That planning effort included the development of a comprehensive Work Breakdown Structure (WBS). The planning team used experience from previous uprate projects to support the development of the WBS. The specific requirements needed to fulfill each work package within that WBS were also defined and assigned in order to ensure that all necessary work was accomplished. Furthermore, the SPU project also incorporated responses to previous NRC Requests for Additional Information (RAIs) that have been issued for other previous uprates.

Furthermore, Westinghouse has addressed the potential effects of the SPU on Nuclear Steam Supply System (NSSS) systems, components, and safety analyses consistent with the Westinghouse methodology established in WCAP-10263 (Reference 3). Since its submittal to the NRC, the WCAP-10263 methodology has been successfully used as the basis for power uprate projects for over 30 pressurized water reactor (PWR) units.

The methodology in WCAP-10263 (Reference 3) establishes the general approach and criteria for uprate projects, including the broad categories that must be addressed, such as NSSS performance parameters, design transients, systems, components, accidents, and nuclear fuel, as well as the interfaces between the NSSS and balance-of-plant (BOP) systems. The methodology includes the use of well-defined analysis input assumptions and parameter values, use of currently approved analytical techniques, and use of currently applicable licensing criteria and standards. A comprehensive engineering review program consistent with the WCAP-10263 (Reference 3) methodology has been performed for IP2 to evaluate the increase in the licensed core power from 3114.4 to 3216 MWt.

1.2.2 References

1. NRC RIS-2002-03, *Guidance on the Content of Measurement Uncertainty Recapture Power Uprate Applications*, January 31, 2002.
2. NRC RS-001 (Draft), *Review Standard for Extended Power Uprates*, December 2002.
3. WCAP-10263, *A Review Plan for Uprating the Licensed Power of a PWR Power Plant*, January 1983.

1.3 Scope Summary and Application Report Structure

In support of the Indian Point Unit 2 (IP2) Stretch Power Uprate (SPU) Project, the following principal organizations have performed major analyses and evaluations to demonstrate that IP2 will remain in compliance with applicable licensing criteria and requirements at the SPU power level.

- Entergy Nuclear Operations, Incorporated (Entergy)
- Westinghouse Electric Company LLC (Westinghouse)
- Stone & Webster (S&W)
- Siemens-Westinghouse Power Corporation (SWPC)

The scope of the above organizations is discussed in the following subsections.

1.3.1 Entergy Nuclear Operations, Incorporated

Entergy Nuclear Operations, Incorporated (Entergy) has extensive experience in owning, managing, and operating nuclear power plants. Entergy has site resources located at the 10 units that it operates and corporate resources located at headquarters in Jackson, Mississippi, and at ENN offices in White Plains, New York. These resources provide significant experience, talent, and oversight that have been applied to ensure that the IP2 SPU meets all NRC requirements. The Entergy SPU Project Team members have more than 200 years of operations, design, licensing, and management experience at nuclear plants. Two members of the team have been licensed as Senior Reactor Operators at Indian Point.

As licensee and operator, Entergy has the overall technical, contractual, and commercial oversight and decision-making responsibility for the IP2 SPU. Entergy is responsible for oversight of the program, and has monitored the performance of its subcontractors and support organizations regarding scope of responsibility, quality of performance, compliance with schedules, and communication among team member organizations. Entergy controlled the progress of the overall project with input from each of the team member organizations. Entergy reviewed and authorized revisions to the project scope and schedule and managed the commercial implications of those changes. Entergy was responsible for contract management with regard to performance of its contractors. In select cases, Entergy provided supporting analysis based on best engineering methods and practices available for use at the time. On technical matters, Entergy consulted with its subcontractors, but had the final authority related to IP2 decisions.

Entergy reviewed results of the analyses, evaluations, and the design of planned plant modifications, and has developed a plan to incorporate them into the IP2 design and licensing basis.

1.3.2 Westinghouse Electric Company LLC

Westinghouse has extensive experience in the design and analysis of Nuclear Steam Supply System (NSSS) systems, including analyses and evaluations for uprates. Westinghouse has performed all of the accident and transient analyses for IP2 since the initial licensing of the plant in 1973. As the IP2 Original Equipment Manufacturer (OEM) NSSS designer and supplier, Westinghouse has extensive historical design documentation and engineering experience applicable to IP2. Westinghouse worked closely with Entergy in the recent past on the Measurement Uncertainty Recapture (MUR) Uprate Program. Therefore, many of the engineers assigned to the uprate project are familiar with the IP2 design and analyses and have worked closely with Entergy plant personnel. The Westinghouse IP2 SPU Project Team members have recent experience in managing power uprate projects as well as significant engineering and licensing experience applicable to IP2.

Westinghouse scope includes all NSSS-related analyses and evaluations, including the NSSS performance parameters, NSSS design transients, NSSS systems and components, design basis accidents (DBAs), NSSS/balance-of-plant (BOP) interface, containment pressure and temperature analyses, and reactor core nuclear fuel.

1.3.3 Stone & Webster

S&W has been in the forefront of nuclear plant uprating, having successfully worked on over 23 plant uprating projects (completed or in progress) within the past 10 years. S&W has prepared implementation plans, design changes, and performed configuration management updates on the majority of these projects. Experience on these uprate projects, along with knowledge of the IP2 design, documentation system, and uprate project requirements has allowed S&W to develop a sound understanding of this project.

S&W has extensive experience in the design and analysis of BOP systems, including analyses and evaluations for uprates. S&W worked closely with Entergy in the recent past on the MUR Uprate Program. Therefore, many of the engineers assigned to the SPU Program are familiar with the IP2 design and analyses, and have worked closely with Entergy plant personnel. The S&W IP2 SPU Project Team members have recent experience in managing power uprate projects as well as significant engineering experience applicable to IP2.

S&W's analyses and evaluations include the BOP systems and components, including radiological and environmental evaluations. S&W also reviewed the effect on station programs.

The BOP scope of work includes engineering and associated review, evaluations, calculations, and analyses required to support the SPU Project at the uprated NSSS power level of 3216 MWt. This work identifies effects and changes required to plant documentation and hardware, and demonstrates that the plant can operate safely, reliably, and meet regulatory requirements.

NSSS/BOP interface data were developed and exchanged among Entergy, Westinghouse, SWPC, and S&W. This information formed the foundation for the BOP reviews, evaluations, calculations, and analyses associated with the following:

- BOP systems and components
- Pipe stress and supports
- Structures
- Electrical
- BOP Instrumentation and controls
- BOP radiological review
- Environmental assessment
- Generic issues and programs
- Plant procedures

1.3.4 Siemens-Westinghouse Power Corporation

The scope of effort performed by SWPC included the engineering study to evaluate the high-pressure and low-pressure turbines for an uprating from 3114.4 MWt to 3216 MWt core power. The turbine missile analysis was evaluated by SWPC. The uprating study for the turbine upgrade configuration demonstrated that the turbine can pass the steam flow required for the targeted thermal cycle and that the components meet SWPC mechanical and electrical design criteria at the new load.

1.3.5 Structure of this Report

This Licensing Amendment Request (LAR) package is structured as follows:

Section 1, Introduction, presents background and general information related to the IP2 SPU Project.

Section 2, NSSS Analysis, presents the primary and secondary system design performance conditions (parameters) that were developed based on the SPU. These design performance conditions form the basis for all of the NSSS analyses and evaluations contained herein.

Section 3, NSSS and Auxiliary Equipment Design Transients, presents the results of evaluations of the design transients and how they accommodate the revised NSSS design conditions.

Sections 4, NSSS Systems, and 5, NSSS Components, present the NSSS systems (for example, safety injection, residual heat removal [RHR], and control systems) and components (for example, reactor vessel, pressurizer, reactor coolant pumps, steam generator, and NSSS auxiliary equipment) analyses, and evaluations completed for the SPU design conditions.

Section 6, Safety Analysis, provides the results of the accident analyses and evaluations performed for the various analyses areas (for example, steam generator tube rupture [SGTR], loss-of-coolant accident [LOCA] non-LOCA accidents and transients, LOCA and main steam line break [MSLB] mass and energy [M&E] releases, and radiological releases).

Section 7, Nuclear Fuel, addresses the effects of the uprate on the fuel and core design.

Section 8, Turbine Island Analysis, addresses the effects of the uprate on the main turbine.

Section 9 BOP Systems addresses the effects of the uprate on the BOP systems.

Section 10, Generic Issues and Programs, addresses the effects of the uprate in the areas of plant programs and operating procedures.

Section 11, Environmental Impacts, addresses the effects of the uprate on the environmental criteria.

The analyses and evaluations described herein demonstrate that all applicable acceptance criteria will continue to be met based on operation at the SPU conditions at 3216 MWt, and that there are no significant hazards related to this power uprate according to the regulatory criteria of 10CFR50.92 (Reference 1).

1.3.6 References

1. 10CFR50.92, *Issuance of Amendment*, March 6, 1986.

1.4 Power Uprate Project Review Process

1.4.1 Input Parameters and Assumptions

Comprehensive analysis input assumption lists were developed at the beginning of the IP2 Stretch Power Uprate (SPU) Program for the various analytical areas within the work scope of the project. These lists were used to identify the input and assumption requirements and to obtain Entergy input data and approval. Entergy performed a review of the values used for the SPU and revalidated the analysis inputs and assumptions provided to Westinghouse, S&W, and SWPC. In conjunction with development of the individual input assumption lists, a consolidated input assumption list was prepared to aid in the identification and control of input data and assumptions and to promote consistency across the various analytical areas within the SPU Project. These input assumption lists have been incorporated into a database for future use by IP2 in managing and controlling analysis inputs and assumptions. Where necessary, follow-up actions have been initiated to update design basis documents to reflect the inputs and assumptions used for the SPU Project.

The power uprate analyses were performed to reflect the as-built and as-operated plant. If plant drawings (as-built) or plant documentation were required to obtain the latest plant information for use in power uprate analyses, they were obtained from Entergy and used as appropriate to obtain the needed information.

1.4.2 Methodology and Computer Codes

1.4.2.1 Nuclear Steam Supply Systems

The methodology used in evaluation of the effect on the Nuclear Steam Supply System (NSSS) has been structured consistent with the methodology established in Westinghouse WCAP-10263, *A Review Plan for Uprating the Licensed Power of a PWR Power Plant*, 1983 (Reference 1). Since submittal of WCAP-10263 to the NRC, the methodology has been used successfully as a basis for power uprate projects on over 33 plants for a total of 1619 MWe of installed capacity. The uprate projects have ranged from a 1.0-percent to a 26.3-percent increase above base licensed power level.

The methodology in WCAP-10263 (Reference 1) established the basis and criteria for power uprate projects, including the broad categories that must be addressed, such as NSSS performance parameters, design transients, systems, components, accidents, and nuclear fuel, as well as the interfaces between NSSS and the balance-of-plant (BOP) fluid systems. Inherent in this methodology are key points that promote correctness, consistency, and licensability. The key points include the use of well-defined analysis input assumptions and parameters values,

use of currently approved analytical techniques (for example, methodologies and computer codes), and use of currently applicable licensing criteria and standards.

The power uprate analyses and evaluations were performed in accordance with Westinghouse quality assurance requirements defined in the Westinghouse Quality Management System procedures, which comply with 10CFR50 Appendix B (Reference 2) criteria. These analyses and evaluations are in conformance with Westinghouse and industry codes, standards, and regulatory requirements applicable to IP2. Assumptions and acceptance criteria are provided in the appropriate sections of this report.

1.4.2.2 Computer Codes

The IP2 SPU analyses and evaluations were performed using currently approved analytical techniques to demonstrate compliance with the licensing criteria and standards that apply to IP2. In performing these analyses, methodologies and principal computer codes were used that are currently approved by the NRC. Such codes and methods have been used for IP2 and the SPU project consistent with any applicable NRC guidelines or limitations. RETRAN has previously been approved by NRC for non-loss-of-coolant accident analyses, has been generically approved in the Safety Evaluation Report (SER) for WCAP-14882-P-A (Reference 3), and is applicable for use at IP2. The use of GOTHIC for main steamline break (MSLB) outside containment compartment analyses for pressure and temperature analyses has not previously been used on the IP2 docket. GOTHIC has previously been approved by NRC for these types of analyses on a number of plant dockets. The STAADIII computer code has not been previously used on IP2 supports analyses. STAADIII is a widely used industry code for analyzing steel structures such as supports. The other principal analytical techniques are the same as those used for current IP2 analyses as described in the IP2 *Updated Final Safety Analysis Report* (UFSAR) (Reference 4) or in the 1.4-percent measurement uncertainty recapture (MUR) uprate License Amendment Request (LAR).

Table 1-1 contains a list of the principal Westinghouse computer codes used in analyses documented in this Licensing Report. Brief descriptions of the computer codes are provided in Table 1-2.

Any computer codes used in the BOP analyses are industry standards or are in compliance with S&W's 10CFR50 Appendix B (Reference 2) program and do not require specific NRC review prior to use. The computer codes used in the BOP sections will be mentioned as a part of the description of the evaluation performed.

1.4.2.3 Balance of Plant

The methodology used for the evaluation of the BOP was the same as used successfully in many other Power Uprate Projects. The BOP systems, structures, and components were evaluated based on the existing design and licensing basis documented in the IP2 UFSAR (Reference 4) and Technical Specification bases. In addition, evaluations were performed to show acceptable capability in areas where the existing documentation did not demonstrate sufficient capability at the SPU conditions. Summary results are provided in Sections 8, 9, and 10 of this report.

1.4.3 References

1. WCAP-10263, *A Review Plan for Uprating the Licensed Power of a PWR Power Plant*, 1983.
2. 10CFR50, *Appendix B, Quality Assurance Criteria for Nuclear Power Plants and Fuel Reprocessing Plants*, December 11, 1996.
3. Safety Evaluation Report (SER) for WCAP-14882-P-A.
4. *Indian Point Nuclear Generating Unit No. 2, Updated Final Safety Analysis Report*, Docket No. 50-247.

1.5 Major Modifications

Reviews, analyses, and evaluations performed for the Indian Point Unit 2 (IP2) Stretch Power Uprate (SPU) have determined that no major modifications are required to accommodate the uprate to 3216 MWt. To provide additional margin for plant operation and equipment lifetime and to optimize operating points, modifications are planned to the following equipment prior to the implementation of the IP2 uprate to 3216 MWt:

- High-pressure turbine steam path
- Moisture separator reheater (MSR)
- First-stage turbine pressure
- Main power transformer
- Iso-Phase bus (IPB) duct

In addition to these noted modifications, minor modifications will be made to instrument meter ranges, and to operating and Emergency Operating Procedures and setpoints.

Also, Entergy has planned other modifications for implementation separate from, but concurrent with the SPU at the start of Cycle 17. These include a replacement of the primary system resistance temperature detectors (RTDs), and a minor structural upgrade to the fuel assemblies planned for the reload region. The various SPU analyses and evaluations described in this report have accounted for these other modifications as necessary.

1.6 Proprietary Information Designations

Westinghouse

There is information contained in this report that Westinghouse considers Westinghouse Proprietary. The specific information is contained within the brackets with designated superscripted letter (a through f), for example:

[Westinghouse Proprietary Information]^{a,c,e}

The reason for marking Westinghouse Proprietary information in this report is so that if any portion of this report is used to prepare documents to be submitted to the NRC (for example, a licensing report), the authors will be aware of exactly which information is proprietary to Westinghouse and can protect the information accordingly. When a licensing report or any other document is submitted to the NRC for review, either the information proprietary to Westinghouse Electric Company LLC must be omitted from the submittal, or a nonproprietary version suitable for public disclosure must also be submitted.

1.7 Conclusions

This report demonstrates that the stretch power uprate (SPU) can be safely implemented at Indian Point Unit 2 (IP2). The analyses and evaluations described herein demonstrate that all applicable acceptance criteria will continue to be met based on operation at the SPU conditions at 3216-MWt core power, and that there are no significant hazards related to this power uprate according to the regulatory criteria of 10CFR50.92 (Reference 1). Specifically, this uprate can be accommodated without a significant increase in the probability or consequences of an accident previously evaluated, without creating the possibility of a new or different type of accident from any accident previously evaluated, and without exceeding any presently existing regulatory limits applicable to the plants, which may cause a significant reduction in the margin of safety.

Furthermore, Entergy has evaluated the capability of IP2 plant systems and components and has determined that, with minor modifications the plant systems and components are capable of safely supporting the subject increase in rated core thermal power.

This IP2 SPU Project document is a summary of how the plant Nuclear Steam Supply System (NSSS) and balance-of-plant (BOP) systems and components, transient and accident analyses, containment and reactor core, and nuclear fuel, have been addressed to support operation at the SPU power at IP2. The results of the NSSS and BOP analyses and evaluations satisfy the project purpose to demonstrate compliance with all applicable licensing criteria and

requirements. Further, the evaluations and analyses have identified the plant modifications required and the operational effects of the SPU. While minor in scope, the plant effects have been properly documented in accordance with plant policy and procedures. This document, in combination with referenced supporting documentation, forms the basis for the IP2 SPU to 3216 MWt.

1.7.1 References

1. 10CFR50.92, *Issuance of Amendment*, March 6, 1986.

Table 1-1
IP2 SPU Project
Westinghouse Computer Codes Used

| Report Section | Analysis | Computer Code ⁽¹⁾ | Previously Used by IP2 or Accepted by NRC |
|----------------|---|--|--|
| 4.3 | Control Systems Operability – Margin-to-Trip Analysis | LOFTRAN (LOFT12) | Yes ⁽²⁾ |
| 5.2 | Reactor Internals | WECAN THRIVE | Yes ⁽²⁾ Yes |
| 5.5 | RCS Piping and Supports | WESTDYN STAAD III | Yes ⁽²⁾ No ⁽³⁾ |
| 5.6 | SG Thermal-Hydraulic | GENF ATHOS | Yes ⁽²⁾ Yes ⁽²⁾ |
| 6.2 | Large-Break Best-Estimate LOCA (LBBELOCA) | WCOBRA/TRAC | Yes ⁽²⁾ |
| | Small-Break LOCA (SBLOCA) | NOTRUMP/ SBLOCTA | Yes ⁽²⁾ Yes ⁽²⁾ |
| 6.3 | Non-LOCA Transients | ANC FACTRAN PHOENIX-P RETRAN TWINKLE VIPRE LOFTRAN | Yes ⁽²⁾ Yes ⁽²⁾ Yes ⁽²⁾ Yes ⁽⁴⁾ Yes ⁽²⁾ Yes ⁽²⁾ Yes ⁽²⁾ |
| 6.4 | SGTR | LOFTTR2 | Yes |
| 6.5 | LOCA M&E LOCA Integrity Inside Containment | SATAN VI WREFLOOD EPITOME FROTH COCO | Yes ⁽²⁾ Yes ⁽²⁾ Yes ⁽²⁾ Yes ⁽²⁾ Yes ⁽²⁾ |
| 6.6 | MSLB Containment Integrity MSLB Inside Containment MSLB Outside Containment | LOFTRAN COCO GOTHIC | Yes ⁽²⁾ Yes ⁽²⁾ Yes ⁽⁵⁾ |
| 6.6 | MSLB M&E | LOFTRAN | Yes ⁽²⁾ |
| 6.7 | LOCA Hydraulic Forces | MULTIFLEX 3.0 LATFORC FORCE 2 THRUST | Yes ⁽²⁾ Yes ⁽²⁾ Yes ⁽²⁾ Yes ⁽²⁾ |

Table 1-1 (Cont'd)
IP2 SPU Project
Westinghouse Computer Codes Used

| Report Section | Analysis | Computer Code ⁽¹⁾ | Previously Used by IP2 or Accepted by NRC |
|----------------|---|------------------------------|--|
| 6.11 | Radiation Source Terms | Origen2.1 | Yes ⁽⁶⁾ |
| 7.1 | Fuel Assemblies | NKMODE WEGAP WECAN | Yes ⁽²⁾ Yes ⁽²⁾ Yes ⁽²⁾ |
| 7.2 | Core Thermal-Hydraulic Design | THING IV VIPRE | Yes ⁽²⁾ Yes ⁽²⁾ |
| 7.3 | Core Design | ANC PHOENIX-P | Yes ⁽²⁾ Yes ⁽²⁾ |
| 7.4 | Fuel Rod Design and Performance | PAD 3.4; PAD 4.0 | Yes ⁽²⁾ |
| 7.5 | Neutron Fluence | DORT/BUGLE-96 | Yes ⁽²⁾ |
| 7.5 | Reactor Internals Heat Generation Rates | DORT/BUGLE-96 | Yes ⁽²⁾ |

Notes:

1. See Table 1-2 for a brief description of each code.
2. Used in IP2 UFSAR or 1.4% uprate LAR.
3. STAAD III is a widely used industry computer code for structural analysis.
4. RETRAN code and methods were generically approved by NRC SER on WCAP-14882-P-A and are applicable for use at IP2.
5. GOTHIC is a widely used industry computer code for containment and compartment analyses. In association with a power uprate, GOTHIC was used to evaluate high-energy line breaks outside containment on the Perry Nuclear Power Plant Power Uprate Submittal (ML0036900960, 3-1-00; ML0037244410, 6-1-00 [SER]).
6. Origen2.1 is a widely-used transport and radiation source term code that is noted as acceptable in RG 1.183.

Table 1-2
Computer Code Description

ANC

ANC is an advanced nodal code capable of two-dimensional and three-dimensional neutronics calculations. ANC is the reference model for certain safety analysis calculations, power distributions, peaking factors, critical boron concentrations, control rod worths, reactivity coefficients, etc. In addition, three-dimensional (3-D) ANC validates one-dimensional and two-dimensional results and provides information about radial (x-y) peaking factors as a function of axial position. It can calculate discrete pin powers from nodal information.

ATHOS

ATHOS is a three-dimensional computer program for computational fluid dynamics (CFD) analysis of steam generators. The ATHOS code was developed under the sponsorship of the Electric Power Research Institute (EPRI).

The ATHOS code consists of geometry pre-processor, ATHOS solution, and post-processor modules. The geometry pre-processor simulates the detailed geometry. This geometry simulation includes the detailed tube layout, tube lane blocks, flow distribution baffle, tube support plates, anti-vibration bars (AVB), and opening of the primary separators. The geometry model links thermally with the primary side coolant flow. This thermal link allows the ATHOS module to calculate heat transfer from the primary coolant flow to the secondary side fluid. Therefore, the ATHOS code will calculate both heat flux and tube wall temperature, in addition to typical parameters such as liquid velocity, vapor velocity, steam quality for a two-phase flow like that in the secondary side of a steam generator.

The ATHOS code for the CFD analysis of steam generators has been verified and qualified by EPRI and Westinghouse. The post-processors can process the large amounts of output from the ATHOS calculation. Their capabilities include: (1) velocity vector plots, and (2) contour plots of thermal hydraulic parameters, such as steam quality, velocity, heat flux, and critical steam quality corresponding to departure from nucleate boiling (DNB).

Table 1-2 (Cont'd)
Computer Code Description

COCO

Calculation of containment pressure and temperature is accomplished by use of the digital computer code COCO. COCO is a mathematical model of a generalized containment. The proper selection of various options in the code allows the creation of a specific model for a particular containment design. The values used in the specific model for different aspects of the containment are derived from plant-specific input data. The COCO code has been used and found acceptable to calculate containment pressure transients for many dry containment plants. Transient phenomena within the RCS affect containment conditions by means of convective mass and energy transport through the pipe break.

For analytical rigor and convenience, the containment air-steam-water mixture is separated into a water (pool) phase and a steam-air phase. Sufficient relationships to describe the transient are provided by the equations of conservation of M&E as applied to each system, together with appropriate boundary conditions. Since thermo-dynamic equations of state and conditions may vary during the transient, the equations have been derived for all possible cases of superheated or saturated steam and subcooled or saturated water. Switching between states is handled automatically by the code.

DORT/BUGLE-96

The DORT discrete ordinates transport module of the DOORS 3.1 code package, in conjunction with the BUGLE-96 cross-section library, is used to determine the neutron flux and gamma-ray heating rate environment. This code and the associated cross-section library have been used by Westinghouse to calculate vessel fluences and reactor internals heating rates for other projects that have been submitted to, and approved by, the NRC. Furthermore, these calculational tools are specified in Regulatory Guide 1.190 for this type of work.

EPITOME (see also SATAN-VI and WREFLOOD)

The EPITOME code continues the post-reflood portion of the transient from the time at which the secondary side equilibrates to containment design pressure until the end of the transient. It also compiles a summary of data on the entire transient, including formal instantaneous M&E release tables, and M&E balance tables with data at critical times. EPITOME is essentially an automated hand calculation.

Table 1-2 (Cont'd)
Computer Code Description

FACTRAN

FACTRAN calculates the transient temperature distribution in a cross-section of a metal-clad UO₂ fuel rod and the transient heat flux at the surface of the cladding, using as input the nuclear power and the time-dependent coolant parameters of pressure, flow, temperature, and density. The code uses a fuel model that simultaneously contains the following features:

- A sufficiently large number of radial space increments to handle fast transients, such as a rod ejection accident.
- Material properties that are functions of temperature and a sophisticated fuel-to-cladding gap heat transfer calculation.
- The necessary calculations to handle post-departure from nucleate boiling (DNB) transients: film boiling heat transfer correlations, Zircaloy-water reaction, and partial melting of the fuel.

FORCE2 (See also MULTIFLEX, LATFORC, and THRUST)

The FORCE2 program calculates the hydraulic forces that the fluid exerts on the vessel internals in the vertical direction by using a detailed geometric description of the vessel components along with the transient pressures, mass velocities, and densities computed by the MULTIFLEX code. The analytical basis for the derivation of the mathematical equations employed in the FORCE2 code is the conservation of linear momentum (one-dimensional [1-D]). Note that the computed vertical forces in the LOCA forces analyses do not include body forces on the vessel internals, such as deadweight or buoyancy. The deadweight and other factors are part of the dynamic system model to which the LOCA forces are provided as an external load. When the vertical forces on the reactor pressure vessel (RPV) internals are calculated, pressure differential forces, flow stagnation on, and unrecoverable orifice losses across, and friction losses on, the individual components are considered. These force types are then summed together, depending upon the significance of each, to yield the total vertical force acting on a given component.

Table 1-2 (Cont'd)
Computer Code Description

FROTH

The FROTH code is used for computing the post-reflood transient. The FROTH code calculates the heat release rates resulting from a two-phase mixture present in the steam generator tubes. The M&E releases that occur during this phase are typically superheated due to the depressurization and equilibration of the broken loop and intact loop steam generators. During this phase of the transient, the RCS has equilibrated with the containment pressure, but the steam generators contain a secondary inventory at an enthalpy that is much higher than the primary side. Therefore, there is a significant amount of reverse heat transfer that occurs. Steam is produced in the core due to core decay heat. For a pump suction break, a two-phase fluid exits the core, flows through the hot legs, and becomes superheated as it passes through the steam generator. Once the broken loop cools, the break flow becomes two phase. During the FROTH calculation ECCS injection is addressed for both the injection phase and the recirculation phase. The FROTH code calculation stops when the secondary side equilibrates to the saturation temperature (T_{sat}) at the containment design pressure, after this point the EPITOME code completes the steam generator depressurization.

GENF

GENF is a computer code developed for the steady-state, thermal-hydraulic analysis of non-preheat type vertical U-tube steam generators. Given the geometric parameters, feedwater temperature, primary side flow rate and pressure, GENF computes the circulation ratio, primary and secondary side pressure drops, secondary coolant mass inventory, stability damping factor, and depending on the mode of calculation chosen, steam pressure, primary temperatures, heat load or size of the tube bundle.

Table 1-2 (Cont'd)
Computer Code Description

GOTHIC

GOTHIC solves the integral form of the conservation equations for mass, momentum, and energy for multi-component, two-phase flow. The conservation equations are solved for three fields; continuous liquid, liquid drops, and the steam/gas phase. The three fields may be in thermal non-equilibrium within the same computational cell. This would allow the modeling of subcooled drops (for example, containment spray) falling through an atmosphere of saturated steam. The gas component of the steam/gas field can comprise up to eight different non-condensable gases with mass balances performed for each component. Relative velocities are calculated for each field, as well as the effects of two-phase slip on pressure drop. Heat transfer among the phases, surfaces, and the fluid are also allowed. The GOTHIC code is capable of performing calculations in three modes. The code can be used in the lumped parameter nodal network mode, the two-dimensional (2-D) finite difference mode, and the 3-D finite difference mode. Each of these modes may be used within the same model.

GOTHIC has been used to study hydrogen distributions, containment and compartment pressure and temperature transients, perform flow-field calculations for particle transport purposes, and surge-line flooding studies for loss of RHR cooling events during shutdown operations. The flexible noding and conservation equation solutions in the code allow its application to a wide variety of problems.

LATFORC (See also MULTIFLEX, FORCE2, and THRUST)

The LATFORC computer code utilizes MULTIFLEX-generated field pressures, together with geometric vessel information (component radial and axial lengths), to determine the horizontal forces on the vessel wall and core barrel. The LATFORC code represents the vessel region with a model that is consistent with the model used in the MULTIFLEX blowdown calculation. The downcomer annulus is subdivided into cylindrical segments, formed by dividing this region into circumferential and axial zones. The results of the MULTIFLEX/LATFORC analysis of the horizontal forces are typically stored on magnetic tape and are calculated for the initial 500 msec of the blowdown transient. These forcing functions serve as required input in determining the resultant mechanical loads on primary equipment and loop supports, vessel internals, and fuel grids.

Table 1-2 (Cont'd)
Computer Code Description

LOFTRAN

The LOFTRAN computer program is used for studies of transient response of a PWR system to specified perturbations in process parameters. LOFTRAN simulates up to four-loop systems by modeling the reactor vessel, hot- and cold-leg piping, steam generators (tube and shell sides), and pressurizer. The pressurizer heaters' spray, relief, and safety valves are also considered in the program. Point model neutron kinetics and reactivity effects of the moderator, fuel, boron, and rods are included. The secondary sides of the steam generators use a homogeneous, saturated mixture for the thermal transients, and a water level correlation for indication and control. The Reactor Protection System (RPS) simulation includes reactor trips on neutron flux, over-power and over-temperature, reactor coolant ΔT , high and low pressure, low flow, and high pressurizer level. Control systems, including rod control, steam dump, feedwater control, and pressurizer pressure controls are also simulated. The Safety Injection System (SIS), including the accumulators, is also modeled. LOFTRAN is a versatile program suited to accident evaluation and control studies as well as parameter sizing. It is also used in performing loss of normal feedwater anticipated transient without scram (ATWS) and loss-of-load ATWS evaluations.

LOFT12 is a single-loop version of LOFTRAN used for symmetric transients. LOFT12 was also used in the previous control systems analysis for IP2.

LOFTTR2 is a version of LOFTRAN used for steam generator tube rupture analyses.

Both single-loop and multi-loop codes have been approved by the NRC.

MULTIFLEX

The analysis for LOCA hydraulic forces used the NRC-approved MULTIFLEX computer code, which is the current Westinghouse analytical tool for analyzing LOCA hydraulic forces. The code was used to generate the transient hydraulic forcing functions on the vessel and internals. This code was previously used for LOCA hydraulic forces analyses.

MULTIFLEX 3.0 is an engineering design tool that is used to analyze the coupled fluid-structural interactions in a PWR system during the transient following a postulated pipe rupture in the main RCS. The thermal-hydraulic portion of the MULTIFLEX code is based on the one-dimensional homogeneous model expressed in a set of mass, momentum, and energy conservation equations. These equations are quasi-linear, first-order, partial differential equations solved by the method of characteristics.

Table 1-2 (Cont'd)
Computer Code Description

The employed numerical method utilizes an explicit time scheme along the respective characteristics. MULTIFLEX considers the interaction of the fluid and structure simultaneously, whereby the mechanical equations of vibration are solved through the use of the modal analysis technique. MULTIFLEX 3.0 generates the input for the post-processing codes LATFORC, FORCE2, and THRUST.

NKMODE

NKMODE is used to establish an equivalent finite element model that will preserve the dynamic properties of the fuel assembly. Parametric studies of the assembly vibrational frequencies and mode shapes are performed using NKMODE. NKMODE calculates a set of equivalent spring-mass elements representing an individual fuel assembly structural system.

NOTRUMP/SBLOCTA

The approved codes for Appendix K small-break LOCA (SBLOCA) analyses are NOTRUMP and SBLOCTA. The NOTRUMP computer code is a state-of-the-art, 1-D general network code consisting of a number of advanced features. Among these features are the calculation of thermal non-equilibrium in all fluid volumes, flow regime-dependent drift flux calculations with counter-current flow limitations, mixture level tracking logic in multiple-stacked fluid nodes, and regime-dependent heat transfer correlations. Additional features of the code include a condensation heat transfer model applied in the steam generator region, a loop seal model, a core reflux model, flow regime mapping, etc.

The SBLOCTA computer code is used to model the fuel rod response to the SBLOCA transient. It models three rods in the hot assembly (hot, average, and adjacent), including simultaneous radial and axial conduction. Other modeling features include various skewed axial power shapes, assembly blockage model due to clad swell, and rupture and zirc/water reaction.

NOTRUMP is used to model the thermal-hydraulic behavior of the system and thereby obtain time-dependent values of various core region parameters, such as system pressure, temperature, fluid levels and flow rates, etc. These are provided as boundary conditions to SBLOCTA. SBLOCTA then uses these conditions and various hot channel inputs to calculate the rod heatup, and ultimately, the peak clad temperature (PCT) for a given transient. Additional variables calculated by SBLOCTA are cladding pressure, strain, and oxidation.

Table 1-2 (Cont'd)
Computer Code Description

ORIGEN2.1

Fission product inventories were modeled with ORIGEN2, Version 2.1. ORIGEN2 is a versatile point-depletion and radioactive-decay computer code for use in simulating nuclear fuel cycles and calculating the nuclide compositions and characteristics of materials contained therein. The ORIGEN2 code is an industry-standard code based on the latest industry experimental data. In general, the data are up to date, well documented, and accepted by the industry. Furthermore, this calculational tool is specified in Regulatory Guide 1.183 for this type of work.

PAD 3.4/4.0

The NRC-approved PAD code, with NRC-approved models for in-reactor behavior, is used to calculate the fuel rod performance over its irradiation history. PAD is the principal design tool for evaluating fuel rod performance. PAD iteratively calculates the interrelated effects of temperature, pressure, clad elastic and plastic behavior, fission gas release, and fuel densification and swelling as a function of time and linear power. Fuel rod design and safety analyses are based on updated values (up to 100-percent helium gas release) for the integral fuel burnable absorber (IFBA) helium gas release model.

PAD is a best-estimate fuel rod performance model, and in most cases the design criterion evaluations are based on a best-estimate-plus-uncertainties approach. A statistical convolution of individual uncertainties due to design model uncertainties and fabrication dimensional tolerances is used. As-built dimensional uncertainties are measured for some critical inputs, for example, fuel pellet diameter, and when available, can be used in lieu of the fabrication uncertainties.

PHOENIX-P

PHOENIX-P is a 2-D, multi-group transport theory computer code. The nuclear cross-section library used by PHOENIX-P contains cross-section data based on a 70-energy-group structure derived from ENDF/B-VI files. PHOENIX-P performs a 2-D, 70-group nodal flux calculation that couples the individual subcell regions (pellet, cladding, and moderator) as well as surrounding rods via a collision probability technique. This 70-group solution is normalized by a coarse energy group flux solution derived from a discrete ordinates calculation. PHOENIX-P is capable of modeling all cell types needed for PWR core design applications.

Table 1-2 (Cont'd)
Computer Code Description

RETRAN

RETRAN is used for studies of transient response of a PWR system to specified perturbations in process parameters. This code simulates a multi-loop system by a lumped parameter model containing the reactor vessel, hot- and cold-leg piping, RCPs, steam generators (tube and shell sides), main steam lines, and the pressurizer. The pressurizer heaters, spray, relief valves, and safety valves may also be modeled. RETRAN includes a point neutron kinetics model and reactivity effects of the moderator, fuel boron, and control rods. The secondary side of the steam generator uses a detailed nodalization for the thermal transients. The RPS simulated in the code includes reactor trips on high neutron flux, overtemperature ΔT (OT ΔT) and overpressure ΔT (OP ΔT), low RCS flow, high- and low-pressurizer pressure, high-pressurizer level, and lo-lo steam generator water level. Control systems are also simulated including rod control and pressurizer pressure control. Parts of the SIS, including the accumulators, may be modeled. RETRAN calculates the transient value of departure from nucleate boiling rate (DNBR) based on input from the core thermal safety limits.

SATAN-VI (See also WREFLOOD and EPITOME)

The SATAN code utilizes the control volume (element) approach with the capability for modeling a large variety of thermal fluid system configurations. The fluid properties are considered uniform, and thermo-dynamic equilibrium is assumed in each element. A point-kinetics model is used with weighted feedback effects. The major feedback effects include moderator density, moderator temperature, and Doppler broadening. A critical flow calculation for subcooled (modified Zaloudek), two-phase (Moody), or superheated break flow is incorporated into the analysis.

Table 1-2 (Cont'd)
Computer Code Description

STAAD-III

STAAD/Pro STAAD-III Revision 2000 is a commercially available computer code suite that is used for analysis and evaluation of supports. The following features of STAAD-III Revision 2000 are used:

- Reaction forces
- Joint deflection
- Member forces
- Member stresses
- Moment at a point in a plane frame with side-sway
- Design of a simple steel frame structure
- Allowable member stresses
- American Institute of Steel Construction (AISC) Code check of member stresses
- Plate element deflections
- Plate element forces
- Plate element stresses
- Response spectra analysis

THINC IV

The THINC-IV computer program is used to determine coolant density, mass velocity, enthalpy, vapor void, static pressure, and DNBR distributions along parallel flow channels within a reactor core under expected steady-state operating conditions. This code has had extensive experimental verification and is considered a best-estimate code. The THINC-IV analysis is based on a knowledge and understanding of the heat transfer and hydro-dynamic behavior of the coolant flow and the mechanical characteristics of the fuel elements. The THINC-IV analysis provides a realistic evaluation of the core performance.

THRIVE

The Thermal Hydraulic Reactor Internals Vessel Evaluation (or THRIVE) code models the reactor vessel and internals system in Westinghouse PWRs and performs the following computations:

- Reactor vessel pressure losses for the thermal design, best estimate, mechanical design, hot-pump overspeed, and cold-full flow rates
- Reactor vessel-internals associated core bypass flows

Table 1-2 (Cont'd)
Computer Code Description

- Reactor internals baffle-barrel region flow rates
- Baffle joint momentum flux and baffle jetting margins of safety
- Baffle plate pressure relief hole velocities
- Reactor internals hydraulic uplift forces
- Hydraulic and geometrical data for use in nuclear safety, fluid systems and reactor internals component analyses

The THRIVE code predicts the RV pressure losses by classical analytical fluid mechanics. THRIVE solves the following continuity and momentum equations for a flow system that represents the entire reactor vessel and internals system:

$$W = \rho VA = \text{constant}$$

$$P_j = P_i + \sum_i^j (K + f/D) \frac{\rho V^2}{2 g_c}$$

Typically, in purely analytical hydraulic analyses, the fluid properties appearing in the equations are known to a high degree of confidence. Therefore, if the pressure loss determination fails, it is generally due to the inability to analytically predict hydraulic loss coefficients of complex geometries. In order to eliminate this potential problem in the THRIVE code, these coefficients were experimentally determined from tests on the 1/7 scale models of the San Onofre, Connecticut Yankee and 3XL pressurized water reactors. Thus, the pressure drops predicted by the code are the results of standard hydraulic methods using coefficients with a sound experimental basis.

THRUST

The THRUST program calculates the hydraulic forces that the fluid exerts on the reactor coolant loop (RCL). The THRUST code uses the MULTIFLEX LOCA pressure transient as input in the calculation of the loop forces. In the THRUST computer code, the loop piping is represented by a series of control volumes. The pressure forces are calculated by THRUST wherever there are changes in either loop area or direction. The LOCA loop forces are then transmitted to the appropriate structural analysis group where these loads are then combined with the other design basis loads (that is, seismic, thermal, and system-shaking loads) for use in qualifying the RCLs under the design basis loads.

Table 1-2 (Cont'd)
Computer Code Description

TWINKLE

TWINKLE is a multi-dimensional spatial neutron kinetics code. The code uses an implicit finite-difference method to solve the two-group transient neutron diffusion equations in one, two, and three dimensions. The code uses six delayed neutron groups and contains a detailed multi-region, fuel-cladding-coolant heat transfer model for calculating point-wise Doppler and moderator feedback effects. The code handles up to 2000 spatial points and performs steady-state initialization. Aside from basic cross-section data and thermal-hydraulic parameters as input, the code accepts basic driving functions such as inlet temperature, pressure, flow, boron concentration, control rod motion, and others. The code provides various output, for example, channel-wise power, axial offset, enthalpy, volumetric surge, point-wise power, and fuel temperatures. It also predicts the kinetic behavior of a reactor for transients that cause a major perturbation in the spatial neutron flux distribution.

VIPRE

VIPRE-01 (VIPRE) is a 3-D subchannel code that has been developed to account for hydraulic and nuclear effects on the enthalpy rise in the core and hot channels. The VIPRE code is based on a knowledge and understanding of the heat transfer and hydro-dynamic behavior of the coolant flow and the mechanical characteristics of the fuel elements. The use of the VIPRE analysis provides a realistic evaluation of the core performance and is used in the thermal-hydraulic analysis.

The VIPRE core model is used with the applicable DNB correlations to determine DNBR distributions along the hot channels of the reactor core under all expected operating conditions. The VIPRE code has been validated against experimental data. The effect of crud on the flow and enthalpy distribution in the core is not directly considered in the VIPRE evaluations. However, conservative treatment by the VIPRE modeling method has been demonstrated to bound this effect in DNBR calculations.

Table 1-2 (Cont'd)
Computer Code Description

WCOBRA/TRAC

WCOBRA/TRAC is a thermal-hydraulic computer code that is used in the approved best-estimate large-break LOCA (BELBLOCA) methodology for the calculation of fluid and thermal conditions in a PWR during a LBLOCA.

WCOBRA/TRAC uses a two-fluid, three-field representation of flow in the vessel component. The three fields are a vapor field, a continuous liquid field, and an entrained liquid drop field. Each field in the vessel uses a set of 3-D continuity, momentum, and energy equations with one exception: a common energy equation is used by both the continuous liquid and the entrained liquid drop fields.

The 1-D components consist of all the major components in the primary system, such as pipes, pumps, valves, steam generators, and the pressurizer. The 1-D components are represented by a two-phase, five-equation, drift flux model. This formulation consists of two equations for the conservation of mass, two equations for the conservation of energy, and a single equation for the conservation of momentum. Closure for the field equations requires specification of the interphase relative velocities, interfacial heat and mass transfer, and other thermo-dynamic and constitutive relationships. This best-estimate computer code contains the following features:

- Ability to model transient 3-D flows in different geometries inside the vessel
- Ability to model thermal and mechanical non-equilibrium between phases
- Ability to mechanistically represent interfacial heat, mass, and momentum transfer in different flow regimes
- Ability to represent important reactor components such as fuel rods, steam generators, RCPs, etc.

WECAN

The WECAN computer code is a general-purpose, finite element code with capabilities including structural and thermal-hydraulic static and dynamic analyses. It is a direct descendent of the mainframe-version of the WECAN code that has been used in the nuclear industry since the early 1970s. It has been used by Westinghouse for safety-related work for many years on essentially all Westinghouse-provided NSSS analyses, such as core structural design (analyses including static, dynamic, and thermal), primary piping, primary equipment supports, primary equipment components, and spent fuel rack design.

Table 1-2 (Cont'd)
Computer Code Description

The WECAN computer program can be used to solve a large variety of structural analysis problems. These problems can be 1-, 2-, or 3-D in nature. It is capable of static elastic and inelastic analysis, steady-state hydraulic analysis, standard and reduced modal analysis, harmonic response analysis, and transient dynamic analysis.

The WECAN program is based on the finite element method of analysis. The analyst must model, or idealize, the structure in terms of discrete elements and apply loadings and boundary conditions to these elements. The stiffness (or conductivity) matrix for each element is assembled into a system of simultaneous linear equations for the entire structure. This set of equations is then solved by a variation of the Gaussian elimination method known as the wave-front technique. This type of solution makes it possible to solve systems with a large number of degrees of freedom using a minimum amount of core storage. The maximum number of allowed degrees of freedom in the wave front depends on the amount of core available, which in turn depends on the type of analysis being performed.

WECAN is organized in such a way that additional structural elements can be added with a minimum of effort. Input formats are similar for all elements and all types of analysis. Input used in the static analysis of a structure can be used for a dynamic analysis with only minor modifications.

WEGAP

WEGAP calculates the dynamic structural response of a PWR core. WEGAP represents the transient structural response of one row of fuel assemblies, including impact at the grid elevation. With the appropriate analysis parameters such as grid impact stiffness and damping, the number of fuel assemblies in a planar array and gap clearance established, the WEGAP reactor core model is used for analyzing transient loadings.

WESTDYN

WESTDYN, a computer program used for the structural analysis of piping systems, calculates displacement, internal forces, and stress distributions in 3-D piping models, while subjecting them to static and dynamic loads.

The static analysis includes pressure, deadweight, thermal expansion, distributed and point loads, anchor motion, and uniformly applied accelerations.

Table 1-2 (Cont'd)
Computer Code Description

The dynamic analysis includes seismic or hydro-dynamic response spectra and time-history dynamic analysis. The time-history dynamic analysis includes options for non-linear supports, support gaps, and unidirectional single acting restraints.

In addition, WESTDYN uses post-processors for the stress analysis of American Society of Mechanical Engineers (ASME) Code Class 1, 2, and 3, or ANSI B31.1 piping, and also for generating support load summary sheets and equipment, and component qualification input data.

WESTDYN automatically calculates stress indices for standard ANSI fittings by user selection of the ASME piping evaluation code and edition. Allowable piping stress limits, coefficients of thermal expansion, and moduli of elasticity for a wide range of materials are also automatically calculated with user-supplied design and operating data.

WREFLOOD (See also SATAN-IV and EPITOME)

The WREFLOOD code is used for computing the reflood transient. It addresses the portion of the LOCA transient where the core reflooding phase occurs after the primary coolant system has depressurized (blowdown) due to the loss of water through the break, and when water supplied by the Emergency Core Cooling System (ECCS) refills the reactor vessel and cools the core.

The WREFLOOD code consists of two basic hydraulic models: one for the contents of the reactor vessel, and one for the coolant loops. The two models are coupled through the interchange of the boundary conditions applied at the vessel outlet nozzles and at the top of the downcomer. Additional transient phenomena, such as pumped safety injection and accumulators, RCP performance, and steam generator releases are included as auxiliary equations that interact with the basic models as required. The WREFLOOD code permits the capability to calculate variations during the core reflooding transient of basic parameters, such as core flooding rate, core downcomer water levels, fluid thermo-dynamic conditions (that is, pressure, enthalpy, density) throughout the primary system, and mass flow rates through the primary system.

2.0 NUCLEAR STEAM SUPPLY SYSTEM ANALYSIS

The SPU Project included Nuclear Steam Supply System (NSSS) performance analyses to develop bounding NSSS Performance Capability Working Group (PCWG) parameters for use in the analyses and evaluations of the NSSS, including parameters for NSSS design transients and analyses of systems, components, accidents, and nuclear fuel.

2.1 Nuclear Steam Supply System Parameters

2.1.1 NSSS Performance Capability Working Group Parameters

The NSSS primary and secondary system design parameters are the fundamental system condition inputs (temperatures, pressures, and flow) that are used as the basis for all of the NSSS analyses and evaluations. They provide the Reactor Coolant System (RCS) and secondary system conditions (temperatures, pressures, flow) that are used as the basis for the design transients and for systems, components, accidents, and fuel analyses and evaluations. Revised design parameters were developed to reflect the increase in the IP2 licensed core power from 3114.4 to 3216 MWt. The parameters for the 3114.4-MWt uprate are shown in Table 2.1-1 (Reference 1). The stretch power uprate (SPU) parameters are shown in Table 2.1-2. As discussed in this report, the parameters in Table 2.1-2 have been reconciled with the applicable systems and components evaluations, as well as safety analyses, performed in support of the SPU.

The PCWG parameters were established using conservative assumptions to provide bounding conditions to be used in the NSSS analyses. For example, the RCS flow assumed in generating the primary and secondary side conditions was the thermal design flow (TDF), which was a conservatively low flow that accounted for flow measurement uncertainty and assumed a steam generator tube plugging (SGTP) level of 10 percent. The resulting primary and secondary-side design conditions will bound actual plant operations at the 3216-MWt uprate power level.

The method and mathematical model used to calculate the IP2 design parameter values in Table 2.1-2 used basic thermal, hydraulic, and engineering principles, including energy and mass balances. The code used to determine the NSSS design parameters is called SGPER (Steam Generator PERFORMANCE). Explicit NRC approval is not needed for SGPER, since it is used to facilitate fundamental engineering calculations that could be performed by hand. The code, method, and mathematical model have been successfully used to support all previous uprates for Westinghouse plants.

2.1.2 Input Parameters and Assumptions

Four cases of design performance parameters were developed for the IP2 SPU to cover combinations of SGTP and T_{avg} operating conditions. The following assumptions were common to all four sets:

- Westinghouse Model 44F steam generators

- Thermal Design flow (TDF) of 80,700 gpm/loop
- NSSS uprated power level of 3216 MWt core power with a conservatively high value of 14 MWt net heat input from the primary Reactor Coolant System (RCS) reactor coolant pumps (RCPs)
- Westinghouse 15 x 15 Vantage+ and upgrade fuel design
- Total design core bypass flow of 6.5 percent that accounts for intermediate flow mixing (IFM) grids and thimble plug removal
- T_{Feed} range of 436.2 to 390°F

2.1.3 Discussion of Parameter Cases

Table 2.1-2 provides the NSSS design parameter cases generated and used as the basis for the SPU Project. Four cases were developed.

The four cases are distinguished as follows:

- Case 1 Presents the parameter values applicable for NSSS system and component analyses and for accident analyses and evaluations. They include reactor vessel T_{avg} of 549°F and 0-percent SGTP.
- Case 2 Presents the parameter values applicable for NSSS system and component analyses and for accident analyses and evaluations. They include reactor vessel T_{avg} of 549°F and 10-percent SGTP.
- Case 3 Presents the parameter values applicable for NSSS system and component analyses and for accident analyses and evaluations. They include reactor vessel T_{avg} of 572°F and 0-percent SGTP.
- Case 4 Presents the parameter values applicable for NSSS system and component analyses and for accident analyses and evaluations. They include reactor vessel T_{avg} of 572°F and 10-percent SGTP.

2.1.4 Acceptance Criterion

The acceptance criterion for the determination of the PCWG parameters was to provide bounding conditions to be used in the NSSS analyses with appropriate levels of conservatism that would also provide Entergy with adequate margin in plant operation.

2.1.5 Results and Conclusions

The resulting PCWG parameters, shown in Table 2.1-2, were used by Westinghouse in the NSSS analytical efforts. Westinghouse performed the analyses and evaluations based on the parameter sets that were most limiting, so that the analyses would support operation over the range of conditions specified.

2.1.6 References

1. Entergy Nuclear Operations, Inc., *Indian Point Nuclear Generating Unit No. 2, 1.4-Percent Measurement Uncertainty Recapture Power Uprate License Amendment Request Package.*

**Table 2.1-1
Design Power Capability Parameters
IP2 3114.4 MWt**

| Thermal Design Parameters | Case 1 | Case 2 | Case 3 | Case 4 |
|---|---------|--------|----------------------|--------|
| NSSS Power % | 100 | 100 | 100 | 100 |
| MWt | 3127 | 3127 | 3127 | 3127 |
| 10 ⁶ Btu/hr | 10,670 | 10,670 | 10,670 | 10,670 |
| Reactor Power MWt ⁽¹⁾ | 3115 | 3115 | 3115 | 3115 |
| 10 ⁶ Btu/hr | 10,629 | 10,629 | 10,629 | 10,629 |
| Thermal Design Flow, loop gpm | 80,700 | 80,700 | 80,700 | 80,700 |
| Reactor 10 ⁶ lb/hr | 126.6 | 126.6 | 126.6 | 126.6 |
| Reactor Coolant Pressure, psia | 2250 | 2250 | 2250 | 2250 |
| Core Bypass, % | 6.5 | 6.5 | 6.5 | 6.5 |
| Reactor Coolant Temperature, °F | | | | |
| Core Outlet | 587.4 | 587.4 | 615.7 | 615.7 |
| Vessel Outlet | 583.0 | 583.0 | 611.7 | 611.7 |
| Core Average | 552.9 | 552.9 | 583.0 | 583.0 |
| Vessel Average | 549.4 | 549.4 | 579.2 | 579.2 |
| Vessel/Core Inlet | 515.8 | 515.8 | 546.7 | 546.7 |
| Steam Generator Outlet | 515.5 | 515.5 | 546.4 | 546.4 |
| Steam Generator | | | | |
| Steam Temperature, °F | 490.4 | 479.6 | 522.6 ⁽²⁾ | 512.0 |
| Steam Pressure, psia | 624 | 564 | 831 ⁽²⁾ | 758 |
| Steam Flow, 10 ⁶ lb/hr total | 13.48 | 13.47 | 13.56 ⁽²⁾ | 13.53 |
| Feed Temperature, °F | 431.8 | 431.8 | 431.8 | 431.8 |
| Moisture, % max. | 0.25 | 0.25 | 0.25 | 0.25 |
| Tube Plugging Level (%) | 0 | 25 | 0 | 25 |
| Zero Load Temperature, °F | 547 | 547 | 547 | 547 |
| HYDRAULIC DESIGN PARAMETERS | | | | |
| Mechanical Design Flow, gpm | 97,700 | | | |
| Minimum Measured Flow, gpm/total | 330,000 | | | |

Notes:

1. Conservatively bounds the MUR uprate value of 3114.4.
2. For analyses limited by high steam pressure, conditions corresponding to a maximum steam pressure of 855 psia, steam temperature of 525.9°F and steam flow of 13.58 x 10⁶ lb/hr are assumed. This covers the possibility that the plant could operate with better-than-expected steam generator performance.

**Table 2.1-2
Design Power Capability Parameters
IP2 3216 MWt (3.26% Uprate)**

| Thermal Design Parameters | Case 1 | Case 2 | Case 3 | Case 4 |
|--|----------------------------|----------------------------|----------------------------|----------------------------|
| NSSS Power % | 100 | 100 | 100 | 100 |
| MWt | 3230 ⁽⁴⁾ | 3230 ⁽⁴⁾ | 3230 ⁽⁴⁾ | 3230 ⁽⁴⁾ |
| 10 ⁶ Btu/hr | 11,021 | 11,021 | 11,021 | 11,021 |
| Reactor Power MWt | 3216 | 3216 | 3216 | 3216 |
| 10 ⁶ Btu/hr | 10,973 | 10,973 | 10,973 | 10,973 |
| Thermal Design Flow, loop gpm | 80,700 | 80,700 | 80,700 | 80,700 |
| Reactor 10 ⁶ lb/hr | 126.8 | 126.8 | 126.8 | 126.8 |
| Reactor Coolant Pressure, psia | 2250 | 2250 | 2250 | 2250 |
| Core Bypass, % | 6.5 ⁽³⁾ | 6.5 ⁽³⁾ | 6.5 ⁽³⁾ | 6.5 ⁽³⁾ |
| Reactor Coolant Temperature, °F | | | | |
| Core Outlet | 588.1 | 588.1 | 610.0 | 610.0 |
| Vessel Outlet | 583.7 | 583.7 | 605.8 | 605.8 |
| Core Average | 552.6 | 552.6 | 575.9 | 575.9 |
| Vessel Average | 549.0 | 549.0 | 572.0 | 572.0 |
| Vessel/Core Inlet ⁽⁷⁾ | 514.3 | 514.3 | 538.2 | 538.2 |
| Steam Generator Outlet | 514.0 ⁽⁶⁾ | 514.0 ⁽⁶⁾ | 537.9 | 537.9 |
| Steam Generator | | | | |
| Steam Temperature, °F | 488.2 | 484.5 | 513.0 ⁽⁵⁾ | 509.4 |
| Steam Pressure, psia | 611 ⁽²⁾ | 590 ⁽²⁾ | 765 ⁽⁵⁾ | 741 |
| Steam Flow, 10 ⁶ lb/hr total | 14.01/13.17 ⁽¹⁾ | 14.00/13.16 ⁽¹⁾ | 14.07/13.22 ⁽¹⁾ | 14.06/13.21 ⁽¹⁾ |
| Feed Temperature, °F | 436.2/390 | 436.2/390 | 436.2/390 | 436.2/390 |
| Moisture, % max. | 0.25 | 0.25 | 0.25 | 0.25 |
| Tube Plugging Level (%) | 0 | 10 | 0 | 10 |
| Zero Load Temperature, °F | 547 | 547 | 547 | 547 |
| HYDRAULIC DESIGN PARAMETERS | | | | |
| Pump Design Point, Flow (gpm)/Head (ft.) | 89,700 / 272 | | | |
| Mechanical Design Flow, gpm | 101,300 | | | |
| Minimum Measured Flow, gpm/total | 348,300 | | | |

Notes:

1. Steam flow is affected by the two different feedwater temperatures.
2. Steam pressure limited to 650 psia to avoid violation of the SG primary to secondary pressure differential limit of 1700 psid.
3. Core bypass flow includes 2.0% due to thimble plug removal and IFMs.
4. RCP heat addition of 14 MWt included.
5. If a high steam pressure is more limiting for analysis purposes, a greater steam pressure of 788 psia, steam temperature of 516.4°F, and steam flow of 14.08x10⁶ lb/hr total should be assumed. This is to cover the possibility that the plant could operate with better than expected steam generator performance.
6. At the component analyst's discretion, the T_{cold} value used in the fatigue stress analysis can remain at the 1.4-percent MUR design transient value of 515.5°F.
7. Actual operation of IP2 is limited to a minimum T_{cold} of 525°F to support the vessel integrity calculations. (See subsection 5.1.2)

3.0 NUCLEAR STEAM SUPPLY SYSTEM AND AUXILIARY EQUIPMENT DESIGN TRANSIENTS

This section discusses the generation of the Nuclear Steam Supply System (NSSS) and auxiliary equipment design transients for the uprated power conditions. Current NSSS design transients were analyzed for their continued applicability at Stretch Power Uprate (SPU) power, and the resulting transient curves were provided to all system and component designers for use in their specific analyses. Section 3.1 describes the evaluation performed. Auxiliary equipment design transients were also evaluated to determine whether they remain applicable for use in the uprating analysis of all the auxiliary equipment in the NSSS. The results of this evaluation are presented in Section 3.2.

As a result of the Indian Point Unit 2 (IP2) SPU Program, the plant operating parameters have changed from the current design parameters. These include parameters that are key for analysis of the NSSS design transients used for component stress analysis of the various NSSS components. The affected parameters are shown in Table 3.1-1, along with the current design values and the SPU Program values. As a result of the parameter revisions, a review of the NSSS design transient effects and the need for a revision of the design transients was performed.

3.1 Nuclear Steam Supply System Design Transients

3.1.1 Introduction

As part of the original design and analyses of the Nuclear Steam Supply System (NSSS) components for Indian Point Unit 2 (IP2), NSSS design transients (that is, temperature and pressure transients) were specified for use in the analyses of the cyclic behavior of the NSSS components. To provide the necessary high degree of integrity for the NSSS components, the transient parameters selected for component stress analyses were based on conservative estimates of the magnitude and frequency of the temperature and pressure transients resulting from various plant operating conditions. The transients selected for use in component stress analyses were representative of operating conditions that could occur during plant operations and were considered to be sufficiently severe or frequent to be of possible significance to component stress analysis. The transients were selected to be conservative representations of transients that, when used as a basis for component stress analysis, would provide confidence that the component was appropriate for its application over the operating life of the plant. For purposes of analysis, the number of transient occurrences was based on an operating life of 40 years.

The NSSS design transients are included in the various component design specifications, and are used to perform stress analyses of the affected components. The existing NSSS design transients for IP2 were developed as part of the 1990 Uprate Program. These transients incorporated a T_{avg} window and encompassed the Model 44F replacement steam generator (RSG). WCAP-11972 (Reference 1) describes these design transients. Reference 2 transmitted the NRC SER for the stretch uprate to 3083.4 MWt NSSS power. These design transients were reviewed for the 1.4-percent Measurement Uncertainty Recapture (MUR) Program and were found to remain valid; no changes to the design transients were made for the 1.4-percent MUR Program (Reference 3). The use of this information in the Stretch Power Uprate (SPU) Program is discussed in subsection 3.1.3.

3.1.2 Input Parameters and Assumptions

NSSS design transients are based primarily on the NSSS design parameters as discussed in Section 2 of this report. The NSSS design parameters, upon which the existing NSSS design transients were based, were compared to the NSSS parameters for the SPU and shown to be different in only a few instances such as steam pressure and feedwater temperature. Because of these differences, the current NSSS design transients were reassessed to determine the adequacy of these design transients for the SPU Program.

3.1.3 Description of Analyses and Evaluations

The design transient applicability evaluation reviewed the differences in the 100-percent power condition design parameter values for the existing plant design conditions, including the SPU and the 1.4-percent MUR in order to establish the design transients for the SPU Program. Table 3.1-1 compares these parameters. The design parameters used in the existing design transient development and for the SPU parameters were compared, and it was concluded that the current design transients remain bracketing and applicable for the SPU. These IP2-specific design transients have been used in the NSSS component stress analyses and evaluations presented in Section 5 of this report.

The NSSS design transients are developed for stress analyses of the various NSSS components. Conservatism is generally included in them via the analysis assumptions associated with either the frequency of occurrence or the transient assumptions. These include:

- Frequencies of occurrence are developed in a conservative fashion. For example, while the plants are operated in a base-loaded fashion, it is assumed that every day a plant loading from 0- to 100-percent power followed by an unloading from 100- to 0-percent power occurs. For the upset transients, it is assumed a reactor trip from 100-percent power occurs 400 times over the plant life (that is, 10 times each year for every year of operation). A loss-of-load is assumed to occur 80 times over the plant life (that is, 2 times each year for 40 years of operation). These transient occurrences are conservative in comparison to actual plant operating experience.
- Conservatisms are taken in the transient analysis assumptions. For example, the normal condition design transients are analyzed assuming they are all at beginning-of-core life (BOL) conditions with conservatively low nuclear reactivity feedback parameters, resulting in the minimum reactivity feedback and maximum parameter (that is, RCS and pressurizer pressure and temperature) transient variations. The loss-of-load transient is analyzed like a conservative anticipated transient without scram (ATWS) event, with no reactivity feedbacks, no credit for any control systems, and no reactor trip until the pressurizer is nearly water solid. The reactor trip transient is assumed to occur at BOL core conditions to result in the minimum decay heat and the maximum RCS cooldown.

The existing design transients have been developed at many different times in the IP2 plant life; some still are the original design transient figures dating back to the original plant design in the 1960s. In addition, the present format for design transient parameter reporting is to show both a transient plotted figure and tabular parameter values. Therefore, even though the existing design transients bracket the SPU Program, all of the design transients were redeveloped

based on the SPU Program design parameters shown in Table 3.1-1 and re-transmitted to the analysts for use in the IP2 SPU Program.

To reduce the analysis effect on the NSSS component stress analysis, the minimum value of T_{cold} used in the various stress analyses was allowed to be limited to the same value used in the existing design transients (that is, a steam generator exit temperature of 515.5°F and a reactor vessel inlet temperature of 515.8°F). Therefore, at the component analyst's discretion, the component stress analysis could be done at the T_{cold} value based on either the SPU Program design value or the existing design transient value.

The SPU Program also includes a feedwater temperature window between 390 and 436.2°F for full-power operating conditions. The existing design transient set does not include feedwater temperature and flow, so no associated design transient revisions are made to reflect the new feedwater temperatures and flows.

3.1.4 Acceptance Criteria

There are no specific acceptance criteria for the design transients. See Section 5 for component criteria.

3.1.5 Results and Conclusions

There are no specific results or conclusions for this section. See Section 5 of this report for component results and conclusions.

3.1.6 References

1. WCAP-11972, *Consolidated Edison Company of New York, Inc., Indian Point Unit 2, NSSS Stretch Rating - 3083.4 MWt Licensing Report*, Rev. 1, March 1989.
2. Letter from D. S. Brinkman (NRC) to S. B. Bram (Consolidated Edison Company), *Issuance of Amendment No. 148, TAC No. 69542, SER for 1990 Stretch Power Upgrading*, March 7, 1990.
3. *Indian Point Nuclear Generating Unit No. 2 1.4-Percent Measurement Uncertainty Recapture Power Uprate License Amendment Request Package*, Entergy Nuclear Operations, Inc., November 2002.

| Table 3.1-1 Operating Conditions for Existing Design Transients vs. SPU Values | | | | |
|---|------------------------------------|-------------|------------------------------------|----------------------|
| | High T_{avg} | | Low T_{avg} | |
| | 1990 Uprate Program ⁽⁴⁾ | SPU Program | 1990 Uprate Program ⁽⁴⁾ | SPU Program |
| T_{hot} , °F | 611.7 | 605.8 | 582.2 | 583.7 |
| T_{cold} , °F ⁽¹⁾ | 547.7 | 537.9 | 515.5 | 514.0 ⁽⁵⁾ |
| T_{steam} , °F ⁽²⁾ | 513.5 | 509.4 | 494.9 ⁽³⁾ | 494.9 ⁽³⁾ |
| P_{steam} , psia ⁽²⁾ | 768 | 741 | 650 ⁽³⁾ | 650 ⁽³⁾ |

Notes:

1. Steam generator outlet; reactor vessel/core inlet is 0.3°F higher.
2. Values are for the maximum steam generator tube plugging (SGTP) condition; these bound the 0% SGTP conditions for design transient development.
3. Values are minimum full-power steam pressure (and corresponding temperature) to avoid violating the steam generator primary-to-secondary pressure differential limit of 1700 psid.
4. These are the values for the design transients developed for the 1990 Uprate Program (Reference 1). These design transients remained valid for the MUR Program.
5. At the component analyst's discretion, the T_{cold} value used in the fatigue stress analysis can remain at the present design transient value of 515.5°F.

3.2 Auxiliary Equipment Design Transients

3.2.1 Introduction

The Indian Point Unit 2 (IP2) auxiliary equipment design specifications included transients that were used to design and analyze the Class 1 auxiliary nozzles connected to the Reactor Coolant System (RCS) and certain Nuclear Steam Supply System (NSSS) auxiliary systems piping, heat exchangers, pumps, and tanks. These transients are described by variations in pressure, fluid temperature, and flow and represent umbrella cases for operational events postulated to occur during the plant lifetime. To a large extent the transients are based on *engineering judgment and experience and are considered to result in parameter changes of such magnitude or to occur frequently enough to be significant in the component design and fatigue evaluation processes.* The transients are sufficiently conservative such that, when used as a basis for component fatigue analysis, they provide confidence that the component will perform as intended over the operating license period of the plant. For purposes of analysis, the number of transient occurrences was based on an operating license period of 40 years.

As part of the IP2 SPU, the auxiliary equipment design transients were reviewed to assess continued applicability.

3.2.2 Input Parameters and Assumptions

The review of the auxiliary equipment design transients was based on the range of NSSS design parameters listed in Table 2.1-2 of this report. The approved range of NSSS design parameters for the SPU were compared with the current NSSS design parameters listed in Table 2.1-1 of this report.

3.2.3 Description of Analyses and Evaluation

An evaluation of the current design transients was performed to determine which transients could be affected by the SPU. The evaluation concluded that the only design transients that could be affected by the SPU are those temperature transients affected by full-load RCS design temperatures.

These temperature transients are defined by the differences between the temperature of the coolant in the (RCS) loops and the temperature of the coolant in the auxiliary systems connected to the RCS loops. The greater the temperature difference the greater the effect these temperature transients have on auxiliary component design and stress evaluation. Since the operating coolant temperatures in the auxiliary systems are not affected by SPU, the

temperature difference between the coolant in the auxiliary systems and the coolant in the RCS loops is only affected by changes in the RCS operating temperatures.

The current design temperature transients are based on a full-load T_{hot} of 630°F and a full-load T_{cold} of 560°F. These full-load temperatures were assumed for equipment design to ensure that the temperature transients would be conservative for a wide range of NSSS design parameters.

3.2.4 Acceptance Criteria and Results

A comparison of the range of NSSS design temperatures for SPU at full-load, that is T_{hot} (583.7 to 605.8°F) and T_{cold} (514.3 to 538.2°F) with the T_{hot} and T_{cold} values used to develop the current design transients indicates that the SPU temperature ranges are lower. These lower full-load operating temperatures result in less severe transients, since the temperature differences are lower between RCS loop temperatures and the lower operating temperatures in the auxiliary systems connected to the RCS. For example, the temperature transients imposed on the Chemical and Volume Control System (CVCS) letdown and charging nozzles associated with starting and stopping letdown and charging flow would be less severe, since the temperature differences are less. Therefore, the current body of auxiliary design transients is conservative for the proposed SPU.

3.2.5 Conclusions

The only auxiliary equipment transients that can be potentially affected by SPU are those temperature transients related to full-load NSSS design temperatures. A review of these temperature transients indicates that if these transients were based on the SPU design parameters they would be less severe. Therefore, the current auxiliary equipment design transients for IP2 remain bounding for the proposed IP2 SPU.

3.2 Auxiliary Equipment Design Transients

3.2.1 Introduction

The Indian Point Unit 2 (IP2) auxiliary equipment design specifications included transients that were used to design and analyze the Class 1 auxiliary nozzles connected to the Reactor Coolant System (RCS) and certain Nuclear Steam Supply System (NSSS) auxiliary systems piping, heat exchangers, pumps, and tanks. These transients are described by variations in pressure, fluid temperature, and flow and represent umbrella cases for operational events postulated to occur during the plant lifetime. To a large extent the transients are based on engineering judgment and experience and are considered to result in parameter changes of such magnitude or to occur frequently enough to be significant in the component design and fatigue evaluation processes. The transients are sufficiently conservative such that, when used as a basis for component fatigue analysis, they provide confidence that the component will perform as intended over the operating license period of the plant. For purposes of analysis, the number of transient occurrences was based on an operating license period of 40 years.

As part of the IP2 SPU, the auxiliary equipment design transients were reviewed to assess continued applicability.

3.2.2 Input Parameters and Assumptions

The review of the auxiliary equipment design transients was based on the range of NSSS design parameters listed in Table 2.1-2 of this report. The approved range of NSSS design parameters for the SPU were compared with the current NSSS design parameters listed in Table 2.1-1 of this report.

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An evaluation of the current design transients was performed to determine which transients could be affected by the SPU. The evaluation concluded that the only design transients that could be affected by the SPU are those temperature transients affected by full-load RCS design temperatures.

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temperature difference between the coolant in the auxiliary systems and the coolant in the RCS loops is only affected by changes in the RCS operating temperatures.

The current design temperature transients are based on a full-load T_{hot} of 630°F and a full-load T_{cold} of 560°F. These full-load temperatures were assumed for equipment design to ensure that the temperature transients would be conservative for a wide range of NSSS design parameters.

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A comparison of the range of NSSS design temperatures for SPU at full-load, that is T_{hot} (583.7 to 605.8°F) and T_{cold} (514.3 to 538.2°F) with the T_{hot} and T_{cold} values used to develop the current design transients indicates that the SPU temperature ranges are lower. These lower full-load operating temperatures result in less severe transients, since the temperature differences are lower between RCS loop temperatures and the lower operating temperatures in the auxiliary systems connected to the RCS. For example, the temperature transients imposed on the Chemical and Volume Control System (CVCS) letdown and charging nozzles associated with starting and stopping letdown and charging flow would be less severe, since the temperature differences are less. Therefore, the current body of auxiliary design transients is conservative for the proposed SPU.

3.2.5 Conclusions

The only auxiliary equipment transients that can be potentially affected by SPU are those temperature transients related to full-load NSSS design temperatures. A review of these temperature transients indicates that if these transients were based on the SPU design parameters they would be less severe. Therefore, the current auxiliary equipment design transients for IP2 remain bounding for the proposed IP2 SPU.

4.0 NUCLEAR STEAM SUPPLY SYSTEM

This section describes the evaluation of the Nuclear Steam Supply System (NSSS) fluid systems that support the Stretch Power Uprate (SPU) Program. Evaluations and analyses were performed to confirm that the NSSS fluid systems continue to perform their intended functions under the SPU conditions. The systems addressed in this section are as follows:

Fluid Systems:

- Reactor Coolant System (RCS)
- Chemical and Volume Control System (CVCS)
- Residual Heat Removal System (RHRS)
- Emergency Core Cooling System (ECCS) (Safety Injection System [SIS]/Containment Spray System [CSS])
- Primary Sampling System (PSS)
- Component Cooling Water System (CCWS)
- Spent Fuel Pit Cooling System (SFPCS)

NSSS/Balance-of-Plant (BOP) Interfaces:

- Main Steam System (MSS)
- Steam Dump System
- Condensate and Feedwater System (C&FS)
- Auxiliary Feedwater System (AFWS)
- Steam Generator Blowdown System (SGBS)

NSSS Control Systems:

- NSSS Control Systems Stability/Operability
- Pressure Control Component Sizing
- Overpressure Protection System (OPS)
- Instrumentation and Control System (I&C)

Results and conclusions are presented within each subsection.

4.1 Nuclear Steam Supply System Fluid Systems

Introduction

This section of the report evaluates the Nuclear Steam Supply System (NSSS) fluid systems for the Indian Point Unit 2 (IP2) SPU conditions. The plant NSSS design data to be evaluated for both the current plant conditions and the SPU power levels are presented in Tables 2.1-1 and 2.1-2, respectively. The data in Table 2.1-2 were evaluated for the SPU Program.

This report section addresses the following NSSS systems:

- Reactor Coolant System (RCS)
- Chemical and Volume Control System (CVCS)
- Residual Heat Removal System (RHRS)
- Emergency Core Cooling System (ECCS)
 - Safety Injection System (SIS)
 - Containment Spray System (CSS)
- Primary Sampling System (PSS)
- Component Cooling Water System (CCWS)
- Spent Fuel Pit Cooling System (SFPCS)

The fluid systems evaluations described in this section were performed at the system level. Evaluations of the NSSS components are described in Sections 5.1 through 5.10 of this report.

4.1.1 Reactor Coolant System

The changes in NSSS design parameters that affect the RCS design bases functions include the increase in core power and the allowable range for average RCS temperature (T_{avg}). Verification that the major RCS components can support these changes is addressed in Sections 5.1 through 5.10 of this report. The increase in core power and the allowable RCS T_{avg} range also affect the duty placed on the RCS control and protection systems. Verification that the RCS control and protection systems can support the SPU is addressed in Sections 4.3 and 6 of this report. This section of the report discusses the RCS fluid system design. The

system design considerations include the pressurizer surge line, safety valves inlet and discharge piping, pressurizer relief tank, power-operated relief valve (PORV) inlet and discharge piping, pressurizer spray subsystem, and RCS instrumentation setpoints (excluding instrument channels used by the control and protection systems).

RCS Design Parameters

The NSSS design parameters at the SPU power level are shown in Table 2.1-2. The revised parameters that affect RCS performance are core power and the resulting full-load T_{cold} and T_{hot} temperatures. The steady-state RCS pressure (2235 psig) and no-load RCS temperature (547°F) have not changed. The changes in full-load RCS temperatures are shown below:

| RCS Temperatures | 1.4-percent MUR Parameters | SPU Parameters |
|---------------------------|----------------------------|------------------|
| T_{cold} (SG Outlet) | 515.5 to 546.4°F | 514 to 537.9°F |
| T_{hot} (Vessel Outlet) | 583.0 to 611.7°F | 583.7 to 605.8°F |

Note that the minimum T_{cold} (SG Outlet) temperature has been limited to 515.5°F to minimize the effects on design transient evaluations. These SPU parameters are based on a T_{avg} window of 549 to 572°F. (The 1.4-percent MUR T_{avg} window was 549.4 to 579.2°F.)

RCS Design Temperature and Pressure

The RCS is specified with a design pressure of 2485 psig and a nominal operating pressure of 2235 psig. The RCS design temperature is 650°F with the exception of the pressurizer, which is designed to 680°F. Based on the SPU RCS parameters, the RCS design pressure and temperature continue to bound the SPU operating conditions. Therefore, it is concluded that the RCS design temperature and pressure are not affected by the SPU conditions, and the design of the RCS pressure boundary is maintained within the original design limits.

The RCS transient operating conditions and associated RCS overpressure evaluations resulting from the RCS and plant transients are discussed in other sections of this report, as follows:

- RCS pressure control via the pressurizer heaters and spray systems, including the capability of the surge line, spray valves, and associated instrumentation and setpoints is discussed in Section 4.3.
- RCS inventory control via the pressurizer level control systems, including the associated instrumentation and setpoints is discussed in Section 4.3.

- RCS temperature control, including the associated instrumentation, is discussed in Section 4.3.
- Protection system actuation, including the associated instrumentation and setpoints, is discussed in Section 6.
- RCS piping analyses, based on the SPU operating conditions, are discussed in Section 5.4.

RCS Heat Capacity

The RCS heat capacity is defined as the amount of heat (in Btus) required to raise or lower the RCS temperature by one degree Fahrenheit (Btu/°F), or, the amount of sensible heat that must be removed or added to the RCS for a given change in RCS temperature. The RCS heat capacity is derived from the composite of the RCS fluid(s) and the component masses. RCS component mass is not changing while the SPU Program change in RCS fluid mass is insignificant. Therefore, the RCS heat capacity is not affected by the SPU.

Reactor Coolant Pump Net Positive Suction Head and Residual Heat Removal Suction Valve Open-Permissive Interlock

This section addresses reactor coolant pump (RCP) net positive suction head (NPSH) and the Residual Heat Removal System (RHRS) suction valves open-permissive interlock, as it relates to RCS flow. Adequate RCP NPSH, at the RCP suction, is monitored by using the RCS wide-range pressure instrument. This same pressure transmitter also provides an input signal to the RHRS suction valves open-permissive interlock. Since the RCS wide-range pressure instrument tap is somewhat removed from the RCP suction point (the wide-range pressure instrument is located in the RCS hot leg), the pressure drop from the RCS wide-range pressure transmitter to the RCP suction must be included when using this instrument for monitoring RCP NPSH. This pressure drop is a function of RCS flow, in addition to other plant physical parameters such as RCS component and piping losses. The RCP NPSH is adequate for the SPU conditions. The RHR system suction valves open-permissive interlock is acceptable for the SPU conditions.

Pressurizer Spray Flow

The pressurizer spray flow is used for RCS pressure control. The driving head for pressurizer spray is the pressure difference from the reactor coolant loop (RCL) spray nozzle to the RCL surge nozzle and is a function of RCS flow and temperature. Since the changes in RCS temperatures are negligible at the SPU conditions, there is no effect on pressurizer spray

performance as a result of the RCS temperature changes at SPU conditions. The RCS flow for the SPU conditions is greater than the flow assumed in the spray performance analysis. Therefore, there is adequate spray flow at the SPU power conditions.

Pressurizer Spray and Surge Line Low-Temperature Alarms

The pressurizer surge line and pressurizer spray line temperature instruments are provided to indicate that the minimum spray and surge line flows are met, so that thermal shock to these lines is minimized when these lines are in use. Since the changes in SPU no-load and minimum full-power RCS hot and cold leg temperatures are very small, the nominal 500°F setpoints of these instruments are not affected by the SPU conditions.

Pressurizer Relief Tank

The pressurizer relief tank (PRT) is designed to accept and quench the design basis discharge from the pressurizer steam space. The PRT is sized to condense and cool a discharge of steam equivalent to 110 percent of the full-power pressurizer steam volume. The amount of energy absorbed by the PRT is related to the volume and pressure of the steam discharged. As indicated in Table 2.1-2, RCS pressure has not changed for the SPU conditions. However, pressurizer level has changed at the SPU conditions.

The sensitivity of the PRT initial water temperature on the PRT design basis performance has previously been evaluated for a 130°F initial PRT water temperature. The maximum containment temperature for SPU analyses is 130°F which results in a 130°F initial PRT water temperature. Acceptable PRT performance was demonstrated for 130°F with the PRT setpoints and parameters validated for the 1.4-percent MUR Program. Therefore, it is concluded that the PRT performance is acceptable with a 130°F initial PRT temperature and assuming the design basis discharge.

The loss-of-load transient associated with the design PRT steam discharge from the pressurizer was reevaluated for the SPU. The pressurizer steam released as a result of this analysis remains bounded by the PRT design conditions described above, including the effects of the maximum PRT temperature of 130°F. Therefore, the PRT is acceptable for the SPU conditions, including the maximum ambient containment temperature to 130°F.

4.1.2 CVCS

The changes in NSSS design parameters that could potentially affect the CVCS design bases functions include the increase in core power and the allowable range for RCS full-load design temperatures. The increase in core power and the allowable range for RCS full-load design

temperatures may also affect the CVCS design bases requirements related to the core re-load boron requirements. Additionally, the allowable range for RCS full-load design temperatures may affect the heat loads that the CVCS heat exchangers must transfer to the CCWS, and in the case of the regenerative heat exchanger, to the charging flow.

Regenerative Heat Exchanger

The regenerative heat exchanger cools the normal letdown flow from the RCS, which is at RCS T_{cold} temperature. The design inlet (RCS T_{cold}) temperature of the regenerative heat exchanger is 554.8°F, which bounds the highest RCS T_{cold} temperature associated with the RCS no-load temperature of 547°F (see Table 2.1-2). The no-load RCS temperature has not changed, while the full load SPU T_{cold} temperature has decreased by a small amount (about 8°F). The performance of the regenerative heat exchanger (that is, less limiting, slightly decreased charging and letdown temperatures) is acceptable at SPU conditions with the minor change in letdown flow (due to the small change in RCS T_{cold} temperature).

Non-Regenerative Heat Exchanger

The non-regenerative heat exchanger cools the letdown flow from the regenerative heat exchanger. Since the change in performance of the regenerative heat exchanger is less limiting at SPU conditions, as discussed in the previous section, there will be a negligible effect on the performance of the non-regenerative heat exchanger. The minor difference in performance (decreased cooling water flow) can easily be accommodated within the capability of the non-regenerative heat exchanger cooling water temperature control valve, TCV-130.

Excess Letdown Heat Exchanger

The excess letdown heat exchanger cools the excess letdown flow from the RCS, which is at RCS T_{cold} temperature. The design inlet (RCS T_{cold}) temperature of the excess letdown heat exchanger is 554.8°F, which bounds the highest RCS T_{cold} temperature associated with the RCS no-load temperature of 547°F. Since the no-load RCS temperature has not changed, and the full-load SPU T_{cold} temperature has decreased by a small amount, the performance of the excess letdown heat exchanger is acceptable at SPU conditions with the change in RCS T_{cold} temperature.

Seal Water Heat Exchanger

The seal water heat exchanger cools the seal return flow from the 4 RCP number 1 seals and the excess letdown flow (from the excess letdown heat exchanger) if it is in service. The RCP heat load (including the thermal barrier heat exchanger) is a function of RCS T_{cold} temperature,

while the excess letdown heat load is a function of excess letdown heat exchanger performance. Since the no-load RCS temperature has not changed, and the full-load SPU T_{cold} temperature has decreased by a small amount, the performance of the seal water heat exchanger is acceptable at SPU conditions with the change in RCS T_{cold} temperature.

Charging, Letdown and RCS Makeup (Boration, Dilution, and N-16 Delay Time)

As discussed in the above sections for the various CVCS heat exchangers, there are negligible effects on their performance at the SPU conditions. Therefore, there will also be negligible effects on the charging (including RCP seal injection) and letdown performance provided by the CVCS. The flow capacity performance of the RCS makeup system is independent of the change in RCS conditions resulting from the SPU conditions. However, the makeup system also relies on storage capacity of various sources of water including primary makeup water and boric acid solutions from both the boric acid storage tanks and the refueling water storage tank (RWST).

Primary makeup water is used to dilute RCS boron, to provide positive reactivity control or to blend concentrated boric acid to match the prevailing RCS boron concentration during RCS inventory makeup operations. Since the flow capacity performance of the RCS makeup system is independent of the change in RCS conditions resulting from the SPU conditions as discussed above, the SPU does not affect the capability of the makeup system to perform these system functions.

The boric acid storage tanks and RWST provide the sources of boric acid for providing negative reactivity control to supplement the reactor control rods. The SPU is expected to have a small effect on the boration requirements that must be provided by the CVCS boration capabilities. The maximum expected RCS boron concentrations are within the capability of the CVCS. The Westinghouse Reload Safety Evaluation (RSE) process (Reference 1) is designed to address boration capability for routine plant changes, such as core reloads, and infrequent plant changes such as a plant uprating that result in a change to core operating conditions and initial core reactivity. Therefore, boration capability will be addressed during the RSE process for each reload cycle.

The letdown flow path is routed inside containment such that there is adequate decay of N-16 before the letdown fluid leaves the containment building. Since the change in letdown flow is negligible, as discussed in the previous paragraphs, this radiation protection feature of the CVCS is not affected by the SPU.

Charging Pump NPSH Due to RWST Temperature Increase from 100°F to 110°F

The RWST can provide emergency makeup supply to the charging pump. As part of the evaluations for the SPU, the assumed RWST maximum temperature has been increased from 100 to 110°F to provide additional margin for operations. Since the maximum RWST temperature is being increased from 100 to 110°F, the charging pump NPSH was evaluated. The limiting NPSH temperature condition for the charging pumps occurs when they are aligned to the volume control tank (VCT), which is at a temperature of 130°F. The NPSH evaluation of charging pump suction from the VCT is not affected by the RWST temperature change, and the 130°F temperature evaluated for the VCT bounds the 110°F RWST temperature considered for the SPU Program. Therefore, the charging pump NPSH previously evaluated bounds charging pump operation when pumping RWST water at 110°F.

4.1.2.1 Primary Chemistry Control

Westinghouse has evaluated the changes in plant parameters as they affect the Primary Chemistry Program for IP2. As noted in the NSSS parameters (see Section 2 of this report), the range of RCS T_{avg} extends from 549 to 572°F; the range of T_{hot} extends from 583.7 to 605.8°F. The best-estimate T_{avg} is expected to be 562°F (Section 7.4). The RWST maximum boron concentration has been increased to 2600 ppm (Section 6). The NSSS parameters (Section 2) for the 1.4-percent MUR provided a range of RCS T_{avg} extending from 549.4 to 579.2°F; the range of T_{hot} extends from 583.0 to 611.7°F. The RWST maximum boron concentration for the 1.4-percent MUR was not specified in the Technical Specifications, but a minimum boron concentration of 2000 ppm was specified (Reference 2). For Improved Technical Specifications (ITS) implementation, a maximum boron concentration of 2500 ppm is specified (Reference 3).

The difference in the RCS temperature ranges for the IP2 SPU from the ranges for the 1.4-percent MUR are slight for the coolant water chemistry. The upper value of T_{avg} is 5.2°F lower, and the upper range of T_{hot} is 5.7°F lower than that for the 1.4-percent MUR.

The chemistry of the NSSS is usually considered to be the chemical composition of the primary coolant and the secondary coolant; the Chemistry Programs are designed to keep concentrations of various chemicals within industry-accepted guidelines. Chemicals present include those purposely added for corrosion and pH control, contaminants, and boric acid added as a chemical shim on the primary side.

As noted above, the IP2 SPU results in relatively small temperature changes in primary and secondary coolant temperatures and these new operating conditions are well within the envelope of conditions used in developing the industry chemistry guidelines. Therefore, the IP2

plant chemistry limits based on industry guidelines are still applicable for the IP2 SPU, and no changes to the Primary Chemistry Program are required for the IP2 SPU.

4.1.3 Residual Heat Removal System

The higher SPU power level results in an increase in the amount of residual heat being generated in the core during normal cooldown, refueling operations and accident conditions. This provides a higher heat load on the residual heat exchangers during the cooldown and also during the refueling outage. The removal of core decay heat for accident conditions is addressed in Section 6 of this report. The increased heat loads will be transferred to the Component Cooling Water System (CCWS) and ultimately to the Service Water System (SWS). Evaluation of the SPU performance of the RHRS in conjunction with the CCWS and SWS with the increased heat loads is addressed in this subsection and in subsections 4.1.6 and 9.6 of this report.

The SPU Program affects the plant cooldown time(s) since core power, and therefore the decay heat increases. The plant cooldown calculation was performed at a core power of 3216 MWt to support the SPU Program. The RCS heat capacity and the other RHR heat loads were explicitly considered in these analyses. The analysis was performed to confirm that the RHR and CCW systems continue to meet their design basis functional requirements and performance criteria for plant cooldown under the SPU conditions.

The following considerations were applied to the SPU cooldown analysis:

- The CCW and RHR heat exchanger data assumes 5-percent tube plugging, as was used for the previous cooldown analyses of record.
- Various CCW system auxiliary heat loads and the RCS heat capacity were included in the normal cooldown case and the Appendix R plant cooldown cases. These heat loads, along with an increase in the spent fuel pit (SFP) heat load (assuming a full SFP of fuel that has operated at 3216 MWt) were used in the cooldown analysis.
- Decay heat curves based on 24 month fuel cycles were used.
- Service water flow rates for Appendix R cooldown were varied to minimize service water flow demand while meeting the Appendix R criteria as shown in Table 4.1-1.

As can be seen from the results summary in Table 4.1-1, the normal plant cooldown time with both trains of CCW and RHR available increased from those for the 1.4-percent MUR. The

primary reason for this is the SPU core power and the corresponding increase in the SFP auxiliary heat load on the CCWS.

The Appendix R cases had a 72-hour time limit for cooldown. For these cases, the minimum CCW heat exchanger service water flow to meet the 72-hour cooldown time limit was determined. In the case considering the SFP heat load (Appendix R, Case 1) the required service water flow rate is 1.510 Mlb/hr or about 3033 gpm per CCW heat exchanger (6066 gpm total). In the case of assuming the SFP heat load is isolated (Appendix R Case 2), the required service water flow is 1.100 Mlb/hr or about 2210 gpm per CCW heat exchanger (4420 gpm total).

4.1.4 Emergency Core Cooling System (Safety Injection System/Containment Spray System)

The required volume, duration, and heat rejection capability of the Safety Injection System (SIS) and Containment Spray System (CSS) flows in the event of a postulated accident were determined based on analytical and empirical models that simulate reactor and containment conditions subsequent to the postulated RCS and Main Steam System (MSS) breaks. As a result of these analyses, the system and component criteria necessary to demonstrate compliance with regulatory requirements at the SPU power level were established. Since the results of these analyses (Section 6) have demonstrated that SIS and CSS provide adequate safety margin, the as-built SIS and CSS are acceptable for the SPU conditions.

The scope of this discussion regarding the ECCS includes the SIS (both low-head and high-head systems) and the CSS performance. Subsequent to ECCS and CSS actuation, the SIS draws water from the RWST during the injection phase and delivers it to the RCS, while the CSS simultaneously draws from the RWST and sprays the containment atmosphere. At the conclusion of RWST draindown, operation of the CSS is terminated. Also at the conclusion of RWST draindown, the SIS is switched to the containment recirculation alignment, drawing fluid from the containment sump. The SIS can also provide recirculation spray to the CSS, if required for continued containment cooling, during the recirculation phase.

Minimum and maximum containment spray flows from the RWST were calculated for the SPU Program. These spray flows were used in the containment and ECCS analyses discussed in Section 6 of this report. Due to RWST boron changes that are also being implemented in conjunction with the SPU Program, the post-LOCA sump pH analysis and required trisodium phosphate (TSP) amount has been recalculated as discussed in subsection 6.11.9.

The plant changes associated with the SPU that affect the hydraulic performance of the high head safety injection (HHSI) system included:

- Increased RWST boron concentration to 2600 ppm
- Increased maximum RWST temperature from 100 to 110°F

The HHSI performance addressed HHSI system input information in support of the SPU ECCS analyses and operation during the recirculation phase for the SPU conditions.

For the SPU Program, the HHSI system analysis addressed the recirculation sump particle and system throttle valve cavitation criteria. The results of this evaluation were acceptable.

The SIS evaluation calculated acceptable maximum low-head safety injection (LHSI) recirculation flows that were revised to account for updated recirculation system performance.

The maximum RWST temperature was changed from 100 to 110°F. The effect of the RWST temperature change on previously calculated flows is negligible since either the new higher temperature was considered, or the temperature at which the volumetric flows are calculated is provided to the analyst to convert the volumetric flow to mass flows. However, the maximum RWST temperature affects NPSH to the safety injection and containment spray pumps, since these pumps take suction from the RWST during the injection phase of SIS and CSS actuation. Therefore, these evaluations addressed the containment spray and RHR pump NPSH when these pumps are aligned to the RWST. The saturation pressure change from 100 to 110°F is: $1.27 - 0.95 \text{ psia} = 0.32 \text{ psia}$ (or a $0.32 \text{ psia} \times 2.3 \text{ ft/psi} = 0.74 \text{ ft}$ decrease in NPSH available).

For the containment spray pumps, the revised NPSH available is still greater than the NPSH required. Therefore, containment spray pump NPSH remains acceptable at 110°F and at SPU conditions.

For the RHR pumps, the limiting NPSH alignment is from the containment sump during recirculation, which is evaluated at 270°F. Therefore, it is concluded that the RHR NPSH evaluation remains bounded by that evaluation for an RWST temperature of 110°F.

For the HHSI pumps, the limiting HHSI pump NPSH conditions occur when the HHSI pumps are aligned to the RWST. The NPSH available is a concern for pump runout conditions. The same HHSI pump runout limits have been maintained for the SPU Program. The effect (due to the RWST temperature change) on the NPSH available is very small and the available NPSH is far greater than the required value. Therefore, it is concluded that the small decrease in NPSH available remains well above the required value. Therefore, the HHSI pumps have adequate NPSH for all analyzed design conditions.

4.1.5 Primary Sampling System

The change in NSSS design parameters that potentially affect the Primary Sampling System (PSS) design bases is the allowable range for average RCS design temperature (T_{avg}). The change in RCS loop operating temperatures may affect the PSS design requirement related to the maximum heat load that the PSS heat exchangers must transfer to the CCWS.

The PSS provides fluid samples from the RCS (pressurizer and hot leg) for laboratory analysis. The sample flows from the RCS are cooled (pressurizer steam samples condensed and cooled) via heat exchangers. Since the SPU alters RCS loop operating temperatures, the PSS heat exchangers were evaluated to assess the effect on the design duty of these heat exchangers.

The scope of this evaluation is limited to the high pressure, remotely obtained samples from the RCS since these sample locations set the limiting process conditions that govern the design of the PSS and associated sample coolers. The PSS is discussed in Section 9.4 of the *IP2 Updated Final Safety Analysis Report (UFSAR)* (Reference 4). The limiting duty for the RCS sample coolers is based on the capability of the cooler to condense and cool a sample stream from the pressurizer steam space. The maximum normal steam condition within the pressurizer is based on the saturation steam temperature (653°F) at normal operating RCS pressure, since the pressurizer is maintained at saturation conditions for RCS pressure control. As discussed in the RCS section above, the RCS operating pressure has not changed at the SPU conditions. Therefore, the design duty of the PSS is not affected as a result of the SPU and the PSS is acceptable for the SPU operating conditions.

4.1.6 Component Cooling Water System

The CCWS is an intermediate system between the various radioactive fluid systems and the Service Water System (SWS). It ensures that leakage of radioactivity from the components being cooled is contained within the plant. Revised heat rejection rates and cooling water flow requirements were assessed for the SPU Program.

Normal Plant Operations (at Power and Refueling)

Section 9.3 of the UFSAR (Reference 4) describes the design bases of the CCWS for IP2. The plant heat loads on the CCWS are as follows:

- Residual heat exchangers
- RCPs
- Non-regenerative heat exchanger
- Excess letdown heat exchanger

- Seal-water heat exchanger
- Sample heat exchangers
- Waste gas compressors
- Reactor vessel support pads
- RHR pumps
- SI pumps
- Recirculation pumps
- SFP heat exchanger
- Charging pumps, fluid drive coolers, and crankcase

Of the CCWS heat loads discussed above, the SFP is the only heat load with a potential to affect the CCWS during normal plant operation. The interaction of the SFPCS and the CCWS is addressed in subsection 4.1.7 for normal plant operation and refueling. All other heat loads are not affected by the SPU during normal (at-power) plant operation. Therefore, it is concluded the CCWS is not affected by the SPU during normal power operation.

Normal and 10CFR50 Appendix R (Fire Protection) Plant Cooldown

The CCWS provides cooling to the RHR heat exchangers during plant cooldown. (See subsection 4.1.3 for discussion of plant cooldown performance.) During plant cooldown, the RHR heat exchanger heat load is controlled by throttling RCS flow so that an acceptable CCWS supply temperature is maintained to the CCWS-serviced equipment. Based on the results of the updated RHR cooldown work described in subsection 4.1.3, the historical CCWS supply temperature limits have been maintained for the SPU. For normal cooldown, the CCWS supply temperature is limited to 120°F while for Appendix R cooldown, the CCWS supply temperature is limited to 125°F. Therefore, the CCWS operation during plant cooldown is acceptable for the SPU Program.

Post-LOCA Plant Cooldown

The CCWS supports post-LOCA ECCS operation during recirculation by providing cooling to the RHR heat exchangers. There could be a small effect (a small increase in sump fluid temperature) during recirculation since decay heat slightly increases with reactor power level. Post-LOCA containment sump temperature performance has been addressed to account for the SPU conditions. Sump temperature at the SPU reactor power remains bounded by the CCW

post-LOCA performance Analysis of Record (AOR). Finally, the overall CCWS performance, which also considers the increased SFP heat load, is summarized below:

| CCW Parameter | Criteria | SPU Results |
|------------------|----------|-------------|
| CCW Pump Suction | 185°F | 152°F |
| CCW Supply | 155°F | 124°F |

Therefore, the post-LOCA CCWS performance is acceptable at the SPU conditions.

4.1.7 SFP Cooling System

The SFP contains spent fuel discharged from the reactor over its operating life. The SPU Program affects the SFPCS performance since core power, and therefore the decay heat of the fuel assemblies increases. Due to the conservatism in the heat load calculations, the assumption of 5-percent plugging of the SFP heat exchanger tubes and the remote probability that the maximum allowable SW and CCW temperatures and would occur simultaneously and coincident with a refueling offload, a cycle-specific heat load evaluation using the anticipated actual conditions at the time of the offload will be performed prior to each refueling outage. This evaluation, based on expected SW temperature, CCW flow, SFP heat exchanger performance capability, supplemental heat removal capability, and reload-specific SFP heat removal requirements will determine the decay time and supplemental cooling capability required such that bulk SFP temperature will remain below 180°F (full core offload).

If the calculation shows that the SFP temperature will exceed 180°F with supplemental cooling, then movement of fuel from the reactor into the SPF will not occur until the fuel has decayed to an acceptable level. The required hold time will be documented in the evaluation. Maintaining the SFP bulk temperature at 180°F or less is consistent with the current operation and design of the SFPCS, as well as the SFP structure itself. Therefore, by administratively controlling the in-core hold time of the fuel after shutdown to ensure that the SFP temperature does not exceed 180°F, it will not be necessary to make physical or analytical modifications to the SFP or its cooling system as a result of the SPU.

Two criteria must be met before spent fuel can be discharged to the SFP:

1. Spent fuel can not be discharged to the spent fuel pit until at least 84 hours after shutdown to satisfy the assumptions of the spent fuel handling accident analysis as discussed in subsection 6.11.9.

2. An additional delay time limit prior to spent fuel discharge is administratively controlled by operating procedures to ensure that the total spent fuel heat load is within the capacity of the spent fuel cooling loop as augmented by supplemental cooling capability to satisfy the bulk pit water temperature limits discussed above. This is a variable time limit primarily dependant upon service water temperature, and cooling capacity with supplemental cooling.

4.1.7.1 Analysis Methods for Reload-Specific SFPCS Capability Calculations

Calculation of Decay Heat Load in SFP

The calculation of the decay heat load on the SFP will be based on the contents of the SFP at the time of the reload. A census of the actual fuel assemblies in the SFP prior to the offload will be used in conjunction with the decay heat characteristics of the fuel to be placed in the SFP from the core. The heat load will be based on decay time, power history, and inventory of the SFP.

Calculation of Heat Removal Capacity

The calculation of heat removal capacity will be based on parameters that affect cooling capability. The specific inputs to the calculation will be chosen to be representative of the conditions predicted to exist at the time the core offload is scheduled to take place. Representative values will be chosen for service water temperature, decay heat load in the SFP, SW and CCW cooling system flow rates, and heat exchanger performance parameters (heat transfer area and tube plugging).

The calculation of supplemental heat removal capacity will be based on the excess cooling needed to keep the SFP temperature below 180°F at the time of planned core offload. Representative values will be chosen for service water temperature, decay heat load in the SFP, SW and CCW cooling system flow rates, and heat exchanger performance parameters (heat transfer area and tube plugging). If the combination of SFPCS capability and supplemental cooling capability is not sufficient, then the planned core offload time will be delayed until the combined capacity is sufficient.

Administrative Controls for SFP Cooling Implementation

Administrative controls for SFP cooling implementation will be included in IP2 procedures.

Adequate Make-Up Supply

The makeup needs have been assessed for normal SFP conditions with a maximum number of fuel assemblies that have been operated. The SFP maximum normal heat load is 17.7 Mbtu/hr. This is based on 20 days elapsed time since the previous shutdown with the maximum number of fuel assemblies in the SFP while still having core offload capacity. If the SFP were to lose all cooling under these conditions with an initial pool temperature of 140°F, the time to boil would be 8.3 hr. The required make-up for boiloff with this heat load would be 35 gpm. Makeup water can be supplied within this time and at this rate from the primary water storage tank, the RWST, or the Fire Protection System.

The core offload refueling heat load was evaluated for SPU conditions to determine the makeup needs. The evaluation assumed a maximum number of fuel assemblies that have been operated at 3216 MWt. With no heat removal by installed or supplemental cooling capability, the time for the spent fuel pit water to rise from 180°F to 212°F is at least 1.67 hours. The maximum required makeup rate for boiloff is 75 gpm (for a full core offload). Makeup water can be supplied within this time and at this rate from the primary water storage tank, the RWST, or the Fire Protection System.

4.1.7.2 Conclusions Regarding Reload-Specific SFPCS Capability Calculations

Because the offload-specific calculations will determine the SFP capability required and such capability will be provided before fuel is offloaded to the SFP, acceptable SFPCS performance will be provided for the SPU conditions. In the event of a total failure of the SFPCS, the spent fuel pool heat inertia will allow sufficient time to place makeup water capability into service. The required SFP makeup capability for the most limiting case requires 75 gpm makeup. The makeup water can be supplied within the required time and at this rate from the primary water storage tank, the RWST, or the Fire Protection System.

4.1.8 References

1. WCAP-9272-P-A, *Westinghouse Reload Safety Evaluation Methodology*, F. M. Bordelon et. al., July 1985.
2. Amendment 238, 11/3/03, *Appendix A to Facility Operating License DPR-26 for Entergy Nuclear Indian Point 2, LLC and Entergy Nuclear Operations, Inc. Indian Point Nuclear Generating Plant Unit No. 2*, Docket No. 50-247 Technical Specifications and Bases, Item Specification 3.6.5.

3. *Amendment 237, 5/22/03, Appendix A to Facility Operating License DPR-26 for Entergy Nuclear Indian Point 2, LLC, and Entergy Nuclear Operations, Inc., Indian Point Nuclear Generating Plant Unit No. 2, Docket No. 50-247 Technical Specifications and Bases, Item 3.3.A.1.a.*
4. *Indian Point Nuclear Generating Unit No. 2, Updated Final Safety Analysis Report, Docket No. 50-247.*

Table 4.1-1

SPU Cooldown Analyses Results

| Case | RHR Cut-in Time (hours after shutdown (ASD)) | Time to Cooldown (hours ASD) | Time to Cooldown in 1.4% MUR Analysis (hours ASD) |
|---|--|--|---|
| 1. Normal Cooldown with CCW Auxiliary Heat Loads | 20.0 | 113.6 (~48 hours to 200°F) | 101.1 (33 hours to 200°F) |
| 2. Appendix R Cooldown Case 1 includes the SFP heat exchanger heat load Case 2 assumes that the SFP heat load is isolated | 28.0 ⁽¹⁾ 30.0 ⁽¹⁾ | 71.9 (SW flow = 1.510 Mlb/hr) 71.9 (SW flow = 1.100 Mlb/hr) | 70.9 |

Note:

1. For Appendix R cooldown, this is the required RHR cut-in time.

4.2 NSSS/Balance-of-Plant Interface Systems

As part of the Indian Point Unit 2 (IP2) Stretch Power Uprate (SPU) Program, the following balance-of-plant (BOP) fluid systems were reviewed to assess compliance with Westinghouse Nuclear Steam Supply Systems (NSSS)/BOP interface guidelines:

- Main Steam System (MSS)
- Steam Dump System
- Condensate and Feedwater System (C&FS)
- Auxiliary Feedwater System (AFWS)
- Steam Generator Blowdown System (SGBS)

The review was based on the range of NSSS design parameters approved for an NSSS power level of 3230 MWt (see Section 2 of this report). The current design parameters are those approved for the 1.4-percent MUR with an NSSS power of 3127 MWt (Reference 1). The various interface systems were reviewed to determine changes to interface information for use in the more detailed BOP analyses discussed in Section 9 of this report.

A comparison of the SPU design parameters (Table 2.1-2) with the current design parameters (Table 2.1-1) previously evaluated for systems and components indicates differences that could affect the performance of the BOP systems.

Evaluations of the above BOP systems relative to compliance with Westinghouse NSSS/BOP interface guidelines were performed to address the NSSS design parameters for SPU that include ranges for parameters such as T_{avg} (549 to 572°F), SGTP (0 to 10 percent), and feedwater temperature (390 to 436.2°F). These ranges on NSSS design parameters result in ranges on BOP parameters such as steam generator outlet pressure (590 to 788 psia) and steam/feedwater mass flow rates (13.16×10^6 lb/hr to 14.08×10^6 lb/hr) (Table 2.1-2). The NSSS/BOP interface evaluations were performed to address the effect of these NSSS design parameters on the BOP. The results of the NSSS/BOP interface evaluations are discussed in the following.

4.2.1 Main Steam System

The following subsections summarize the evaluation of the NSSS interface on the MSS major components relative to the SPU Parameters. The major components of the MSS are the steam generator main steam safety valves (MSSVs), the steam generator power-operated atmospheric relief valves (ARVs) and the main steam isolation valves (MSIVs) and non-return valves.

4.2.1.1 Steam Generator Main Steam Safety Valves

The setpoints of the MSSVs based on the design pressure of the steam generators (1085 psig) and the requirements of the *ASME Boiler and Pressure Vessel (B&PV) Code* (Reference 2). Since the design pressure of the steam generator has not changed for SPU, there is no need to revise the setpoints of the safety valves.

The MSSVs must have sufficient capacity so that main steam pressure does not exceed 110 percent of the steam generator shell-side design pressure (the maximum pressure allowed by the ASME B&PV Code) for the worst-case loss-of-heat-sink event (Reference 3). Based on this requirement, Westinghouse applies the conservative criterion that the valves should be sized to relieve 100 percent of the maximum calculated steam flow at an accumulation pressure not exceeding 110 percent of the MSS design pressure.

IP2 has 20 safety valves with a total rated capacity of 15.108×10^6 lb/hr, which provides about 107.3 percent of the maximum SPU full-load steam flow of the 14.08×10^6 lb/hr (see Table 2.1-2). Therefore, based on the range of NSSS design parameters for the SPU, the capacity of the installed MSSVs meets the Westinghouse sizing criterion.

The original design requirements for the MSSVs (as well as the ARVs and steam dump valves) included a maximum flow limit per valve of 890,000 lb/hr at 1085 psig. Since the actual capacity of any single MSSV, ARV, or steam dump valve is less than the maximum flow limit per valve, the maximum capacity criteria are satisfied.

4.2.1.2 Steam Generator Power-Operated ARVs

The ARVs, which are located upstream of the main steam isolation valves (MSIVs) and adjacent to the MSSVs, are automatically controlled by steam line pressure during plant operations. The ARVs automatically modulate open and exhaust to atmosphere whenever the steam line pressure exceeds a predetermined setpoint to minimize safety valve lifting during steam pressure transients. As the steam line pressure decreases, the ARVs modulate closed and reseal at a pressure below the opening pressure. The ARV set pressure for these operations is between zero-load steam pressure and the setpoint of the lowest set MSSVs. Since neither of these pressures changes for the proposed range of NSSS design parameters, there is no need to change the ARV setpoint.

The primary function of the ARVs is to provide a means for decay heat removal and plant cooldown by discharging steam to the atmosphere when the condenser, the condenser circulating water pumps, or steam dump to the condenser is not available. Under such circumstances, the ARVs, in conjunction with the AFWS, permit the plant to be cooled down

from the pressure setpoint of the lowest set MSSVs to the point at which the Residual Heat Removal System (RHRS) can be placed in service. During cooldown, the ARVs are either automatically or manually controlled. In automatic, each ARV proportional and integral (P&I) controller compares steam line pressure to the pressure setpoint, which is manually set by the plant operator.

In the event of a tube rupture event in conjunction with loss-of-offsite power (LOOP), the ARVs are used to cool down the RCS to a temperature that permits equalization of the primary and secondary pressures at a pressure below the lowest set MSSV. RCS cooldown and depressurization are required to preclude steam generator overfill and to terminate activity release to the atmosphere (Reference 3 and Section 6.4).

The steam generator ARVs are sized to have a capacity equal to about 10 percent of rated steam flow at no-load pressure. This capacity permits a plant cooldown to RHRS operating conditions (350°F) in 4 hours (at a rate of about 50°F/hr), assuming cooldown starts 2 hours after reactor shutdown. This sizing is compatible with normal cooldown capability and minimizes the water supply required by the AFWS. This design basis is limiting with respect to sizing the ARVs, and bounds the capacity required for tube rupture.

An evaluation of the installed capacity (1,369,000 lb/hr at 1020 psia) indicates that the original design bases in terms of plant cooldown capability can still be achieved for the range of SPU NSSS design parameters.

4.2.1.3 MSIVs, MSIV Bypass Valves, and Non-Return Valves

The MSIVs and non-return valves are located outside the containment and downstream of the MSSVs and ARVs. The valves function to prevent the uncontrolled blowdown of more than 1 steam generator and to minimize the RCS cooldown and containment pressure to within acceptable limits following a main steamline break (MSLB). To accomplish this function, the design requirements specified that the MSIVs must be capable of closure within 5 seconds of receiving a closure signal against steam break flow conditions in the forward direction.

Rapid closure of the MSIVs and non-return valves following postulated steam line breaks causes a significant differential pressure across the valve seats and a thrust load on the MSS piping and piping supports in the area of the MSIVs and non-return valves. The worst cases for differential pressure increase and thrust loads are controlled by the steam line break area (that is, mass flow rate and moisture content), throat area of the steam generator flow restrictors, valve seat bore, and no-load operating pressure. Since the SPU does not affect these variables, the design loads and associated stresses resulting from rapid closure of the MSIVs

and non-return valves will not change. Consequently, SPU has no significant effect on the interface requirements for the MSIVs and non-return valves.

The MSIV bypass valves are used to warm up the main steam lines and equalize pressure across the MSIVs prior to opening the MSIVs. The MSIV bypass valves perform their function at no-load and low-power conditions at which the SPU has no significant effect on main steam conditions (for example, steam flow and steam pressure). Consequently, the SPU has no significant effect on the interface requirements for the MSIV bypass valves.

4.2.2 Steam Dump System

The NSSS Reactor Control Systems and the associated equipment (pumps, valves, heaters, control rods, etc.) are designed to provide satisfactory operation (automatic in the range of 15- to 100-percent power) without reactor trip when subjected to the following load transients:

- Loading at 5 percent of full power per minute with automatic reactor control
- Unloading at 5 percent of full power per minute with automatic reactor control
- Instantaneous load transients of plus or minus 10 percent of full power (not exceeding full power) with automatic reactor control
- Load reductions of 50 percent of full power with automatic reactor control and steam dump

The Steam Dump System creates an artificial steam load by dumping steam from ahead of the turbine valves to the main condenser. The Westinghouse sizing criterion recommends that the Steam Dump System (valves and pipe) be capable of discharging 40 percent of the rated steam flow at full-load steam pressure to permit the NSSS to withstand an external load reduction of up to 50 percent of plant-rated electrical load without a reactor trip. To prevent a trip, this transient requires all NSSS Control Systems to be in automatic, including the Rod Control System, which accommodates 10 percent of the load reduction. A steam dump capacity of 40 percent of rated steam flow at full-load steam pressure also prevents MSSV lifting following a reactor trip from full power.

4.2.2.1 Steam Dump System Major Components

IP2 is equipped with 12 condenser steam dump valves and each valve is specified to have a flow capacity of 505,000 lbm/hr at a valve inlet pressure of 650 psia. For the current design parameters that limit the minimum allowable full-load steam pressure to ≥ 650 psia (due to

steam generator tubesheet ΔP limits), steam dump capacity was reported to be adequate for an external load reduction of up to 50 percent of plant rated electrical load (Reference 1).

The capacity of the Steam Dump System (as a percentage of full-load steam flow) decreases as full-load steam pressure decreases and full-load steam flow increases. NSSS operation within the proposed range of design parameters for the SPU Program will result in a decrease in steam dump capability due to increased steam flow. The current minimum allowable full-load steam pressure (650 psia), based on a steam generator tubesheet ΔP limits, is not expected to change. The steam dump capacity would be reduced to 34.4 percent of rated steam flow (14.02×10^6 lb/hr), or 4.816×10^6 lb/hr at a full-load steam pressure of 650 psia. At full-load steam pressures higher than 650 psia, steam dump capacity would increase. For example, at a full-load steam pressure of 788 psia, steam dump capacity would be 44.1 percent of rated flow (13.22×10^6 lb/hr), or 5.835×10^6 lb/hr.

The NSSS control systems margin to trip analysis (Section 4.3 of this report) provides an evaluation of the adequacy of the Steam Dump System in conjunction with the control system setpoints at SPU conditions.

The condenser steam dump valves have NSSS requirements on time for opening and for modulating steam flow. To provide effective control of flow on large step-load reductions or plant trip, the steam dump valves are required to go from full-closed to full-open in 3 seconds at any pressure between 50 psi less than full-load pressure and steam generator design pressure. The dump valves are also required to modulate to control flow. For modulating steam dump flow, the positioning response may be slower with an allowed maximum full stroke time of 20 seconds. These time response requirements are not affected by the SPU and must still be met.

4.2.3 Condensate and Feedwater System

The C&FS must automatically maintain steam generator water levels during steady-state and transient operations. The range of NSSS design parameters will affect both feedwater volumetric flow and system pressure drop. The volumetric flow may increase by as much as 4.4 percent, or decrease by as much as 5.6 percent and, therefore, system pressure drop may increase by as much as 8.5 percent, or decrease by as much as 8.0 percent during full-power operation. Comparison of the SPU design parameters with the 1.4-percent MUR design parameters indicated that steam generator full-power operating pressure may decrease by as much as 67 psi (855 to 788 psia).

The major components of the C&FS are the main feedwater regulator valves (FRVs), bypass feedwater regulator valves (BFRVs), and the C&FS pumps. Each of these major components is discussed in the sections that follow.

4.2.3.1 Main Feedwater Isolation/Feedwater Regulator Valves/Bypass Feedwater Regulator Valves

The main FRVs and BFRVs are located outside containment. The valves function in conjunction with backup trip signals to the feedwater pump discharge isolation valves, feedwater pumps, and other miscellaneous valves to provide redundant isolation of feedwater flow to the steam generators following a steam line break or a malfunction in the steam generator level control system. Isolation of feedwater flow is required to prevent containment overpressurization and excessive RCS cooldowns. Redundant main feedwater isolation is provided by:

- Closure of all the main FRVs and closure of the low-flow feedwater bypass valves, or
- Closure of the main feedwater pump discharge valves that initiate closure of the MFW isolation valves and a trip of the main feedwater pumps.

The quick-closure requirements imposed on the FRVs, BFRVs, and the backup feedwater pump discharge isolation valves causes dynamic pressure changes that may be of large magnitude and must be considered in the design of the valves and associated piping. The worst loads occur following a steam line break from no-load conditions with the conservative assumption that all feedwater pumps are in service providing maximum flow following the break. Since these conservative assumptions are not affected by the SPU, the design loads and associated stresses resulting from rapid closure of these valves will not change.

4.2.3.2 FRVs, C&FS Pumps

The C&FS available head in conjunction with the FRV characteristics must provide sufficient margin for feed control to ensure adequate flow to the steam generators during steady-state and transient operation. A continuous steady feed flow should be maintained at all secondary system loads. To ensure stable feedwater control with variable speed feedwater pumps, the pressure drop across the FRVs at rated flow (100-percent power) should be approximately equal to the dynamic losses from the feed pump discharge to the steam generator. These dynamic losses include the frictional resistance of feed piping, high-pressure feedwater heaters, feed flow meter, and steam generator. To preclude reactor trip following load rejection, adequate margin should be available in the FRVs at full-load conditions to permit C&FS delivery of 96 percent of rated flow with a 100-psi pressure increase above the full-load pressure with

the FRVs fully open (Reference 1). The current Feedwater Pump Speed Control Program is set to provide an FRV pressure drop of approximately 166 psi at full-load, and this pressure drop results in an FRV lift of about 81 percent.

The hydraulic evaluation of the C&FS (refer to Section 9.4) for the range of design parameters approved for the SPU indicates the lift of the FRVs at full power will increase by as much as 5.1 percent (from 81 to 86.1 percent at T_{avg} of 549°F) with the present Feedwater Pump Speed Control Program.

The hydraulic evaluation of the C&FS also concluded that the C&FS could maintain adequate feedwater pump suction pressure, assuming 1 drain tank pump remains in service following a large load rejection.

To provide effective control of flow during normal operation, the FRVs are required to stroke open or closed in 20 seconds over the anticipated inlet pressure control range (approximately 0 to 1600 psig). Additionally, rapid closure of the FRVs is required after receiving a trip close signal in order to mitigate certain transients and accidents. These requirements are not affected by the SPU.

4.2.4 Auxiliary Feedwater System

The AFWS supplies feedwater to the secondary side of the steam generators at times when the normal feedwater system is not available, thereby maintaining the steam generator heat sink. The system provides feedwater to the steam generators during normal unit startup, hot standby, and cooldown operations and also functions as an engineered safety feature (ESF). In the latter function, the AFWS is required to prevent core damage and system overpressurization during transients and accidents, such as a loss of normal feedwater or a secondary system pipe break. The minimum flow requirements of the AFWS are dictated by accident analyses, and since the SPU affects these analyses, evaluations of the limiting transients and accidents are performed to confirm that the AFWS performance is acceptable at the SPU conditions. These evaluations are described in Section 6 of this report and show acceptable results.

4.2.4.1 AFW Storage Requirements

The AFWS pumps are normally aligned to take suction from the condensate storage tank (CST). To fulfill the ESF design functions, sufficient feedwater must be available during transient or accident conditions to enable the plant to be placed in a safe shutdown condition.

The limiting transient with respect to CST inventory requirements is the LOOP transient. The IP2 licensing basis requires that, in the event of a LOOP, sufficient CST useable inventory must

be available to bring the unit from full-power to hot-standby conditions, and maintain the plant at hot standby for 24 hours.

Since the required CST inventory is a function of plant-rated power and other NSSS design parameters, a new analysis was performed to determine the required inventory for the range of NSSS design parameters approved for SPU. This analysis is based on the following conservative assumptions:

- Reactor trip occurs from 102 percent of rated core power (3216 MWt), from a low-low water level in the steam generators. A 2-second delay is assumed before reactor trip following LOOP.
- Steam is released from the steam generators at the first safety valve setpoint plus setting tolerance for drift.
- The steam generators are filled back up to 52-percent narrow range water level.
- The CST operating fluid temperature is at the maximum allowable value (120°F).

The analysis concluded that a minimum required useable inventory of 291,381 gallons is required to meet the plant licensing bases for the range of NSSS design parameters approved for SPU. The CST Technical Specification requirement of 360,000 gallons ensures a usable volume of 291,381 gallons.

4.2.5 Steam Generator Blowdown System

The Steam Generator Blowdown System (SGBS) is used to control the chemical composition of the steam generator secondary side water within the specified limits. The SGBS also controls the buildup of solids in the steam generator secondary.

The blowdown flow rates required during plant operation are based on chemistry control and tube-sheet sweep requirements to control the buildup of solids. The blowdown flow rate required to control chemistry and the buildup of solids in the steam generators is based on allowable condenser in-leakage, total dissolved solids in the plant circulating water, and the allowable primary to secondary leakage. Since these variables are not affected by the SPU, the blowdown required to control secondary chemistry and steam generator solids will not be affected by the SPU.

The inlet pressure to the SGBS varies with steam generator operating pressure. Therefore, as steam generator full-load operating pressure decreases, the inlet pressure to the SGBS control

valves decreases and the valves must open to maintain the required blowdown flow rate into the system flash tank. The 1.4-percent MUR NSSS design parameters (Table 2.1-1) permit a maximum decrease in steam pressure from no-load to full-load of 370 psi (that is, from 1020 to 650 psia). Based on the revised range of SPU NSSS design parameters (Table 2.1-2), the no-load steam pressure (1020 psia) remains the same, and the current minimum allowable full-load steam pressure (650 psia) due to steam generator tubesheet ΔP limits does not change. Therefore, the range of design parameters approved for the SPU will not affect blowdown flow capability.

4.2.6 Conclusions

The following is a brief summary of the NSSS/BOP interface evaluation conclusions for the IP2 SPU Program.

Main Steam System

- The capacity of the installed MSSVs is adequate to meet the original sizing bases for the approved range of NSSS design parameters.
- The capacity of the installed ARVs is adequate to meet the original sizing bases for the approved range of NSSS design parameters.
- SPU does not adversely affect the criteria for the MSIVs and MSIV bypass valves.

Steam Dump System

An evaluation of the Steam Dump System indicates that the minimum system capacity is approximately 34 percent of the SPU full-load steam flow at the current minimum allowable full-load steam pressure of 650 psia. At full-load steam pressures higher than 650 psia, steam dump capacity would increase. The control system's margin to trip analysis provides an evaluation of the adequacy of steam dump in conjunction with the control system setpoints (see Section 4.3 of this report).

Condensate and Feedwater System

- The lift of the FRVs at full power will increase by as much as 5.1 percent (from 81 to 86.1 percent at T_{avg} of 549°F) with the present Feedwater Pump Speed Control Program.
- Per Section 9.4, feedwater pump suction pressure is adequate, assuming 1 drain tank pump remains in service following a large load rejection.

Auxiliary Feedwater System

- The AFWS is capable of delivering the minimum flow requirements for the SPU (see Section 6).
- The CST minimum useable inventory of 291,381 gallons is required to meet the plant licensing bases for the range of NSSS design parameters approved for SPU. The Technical Specification value of 360,000 gallons ensures a usable volume of 291,381 gallons.

Steam Generator Blowdown System

- The blowdown flow required to control secondary chemistry and steam generator solids is not affected by the SPU.
- The NSSS design parameters approved for the SPU coupled with the current minimum allowable full-load steam pressure will not affect blowdown flow capability.

4.2.7 References

1. *Indian Point Nuclear Generating Unit No. 2 1.4-Percent Measurement Uncertainty Recapture Power Uprate License Amendment Request Package*, Entergy Nuclear Operations, Inc., November 2002.
2. *ASME Boiler and Pressure Vessel Code*, Section III, "Rules for Construction of Nuclear Vessels," 1965 Edition with Winter 1965 Addenda, The American Society of Mechanical Engineers, New York, NY.
3. *Indian Point Nuclear Generating Unit No. 2, Updated Final Safety Analysis Report*, Docket No. 50-247.

4.3 Nuclear Steam Supply System Control Systems

4.3.1 NSSS Stability and Operability

4.3.1.1 Introduction

Control systems operability analyses were performed on the Nuclear Steam Supply System (NSSS) control system setpoints for the Indian Point Unit 2 (IP2) plant to determine that there is adequate margin to relevant reactor trip and Engineered Safety Features (ESFs) actuation setpoints for the proposed Stretch Power Uprate (SPU) Program. The conditions that were used as starting points for these analyses are provided in Section 2 of this report (NSSS parameters) and encompass a range of plant operating conditions.

The following cases, at both high- and low- T_{avg} conditions, were analyzed:

- Fifty-percent load rejection from 100-percent power
- Ten-percent step-load decrease from 100-percent power
- Ten-percent step-load increase from 90-percent power
- Turbine trip without reactor trip

4.3.1.2 Input Parameters and Assumptions

The conditions that were used as starting points for these analyses are provided in Section 2 of this report and encompass a range of plant operating conditions. However, the steam pressure for the low T_{avg} conditions shown in Section 2 was not able to be supported by the NSSS design transient analyses described in Section 3.1 of this report. The minimum full-power steam pressure that could be supported was a value of 650 psia (due to steam generator tubesheet ΔP considerations). This resulted in the following full-power T_{avg} values for this minimum acceptable full-power steam pressure:

0-percent tube plugging: Full-power $T_{avg} = 550.5^{\circ}\text{F}$

10-percent tube plugging: Full-power $T_{avg} = 559.5^{\circ}\text{F}$

The stability and operability analyses bracketed all operating conditions: full-power T_{avg} ranging from the above minimum values for a minimum full-power steam pressure of 650 psia to an upper limit of 572.0°F, and 0- to 10-percent steam generator tube plugging (SGTP) levels. The following assumptions were made for all normal transients analyzed:

- All applicable NSSS control systems were assumed to be operational and in the automatic mode of control (that is, rod control, steam dump control, pressurizer level, steam generator level control, and pressurizer pressure control).
- Two-percent initial power level uncertainty was assumed. The remainder of the plant parameters (that is, Reactor Coolant System [RCS] T_{avg} , pressurizer pressure, pressurizer level, steam generator level) were assumed to be at their nominal control system setpoints.
- Best-estimate reactor kinetics parameters were modeled (that is, rod worth, moderator temperature coefficient, Doppler power defect, etc.) Since beginning-of-life (BOL) core physics parameters have lower differential rod worth and a less negative moderator temperature coefficient, modeling BOL core characteristics typically yielded more conservative results that bound the full cycle of operation.
- In general, analysis of 10-percent tube plugging conditions bounds the 0-percent tube plugging conditions. Higher tube plugging was somewhat more conservative for short-term heatup transients due to a slower rate of heat transfer from the primary to secondary side of the plant. Furthermore, lower nominal steam temperatures and pressures reduced steam dump capacity during heatup transients, and reduced margin to safety injection (SI) actuation on low steam pressure during cooldown transients.
- The transient simulations were modeled to run for a 300-second interval (5 minutes). Most challenges to the reactor trip and ESF actuation setpoints occurred within the first minute of the design basis normal condition transients, therefore, this simulation time frame was considered more than adequate for assessing control system response and stability considerations.
- The following protection systems functions have the greatest potential for being challenged during these operability transients and therefore were considered in this analysis (other protection systems would only be challenged during these transients if one of the following did not function).

Overtemperature ΔT

$$\Delta T \left[\frac{1}{(1 + \tau_4 s)} \right] \leq \Delta T_0 \{ K_1 - K_2 \left[\frac{(1 + \tau_1 s)}{(1 + \tau_2 s)} \right] \left(T \left[\frac{1}{(1 + \tau_5 s)} \right] - T' \right) + K_3 (P - P') - f_1(\Delta I) \}$$

| Parameter | Setpoint |
|-----------------|--|
| K_1 | 1.22 |
| K_2 | 0.020/°F |
| K_3 | 0.0007/psig |
| τ_1 | 25 sec |
| τ_2 | 3 sec |
| τ_4 | 4 sec |
| τ_5 | 4 sec |
| ΔT_0 | Indicated ΔT at rated thermal power, °F |
| T | Measured RCS T_{avg} , °F |
| T' | Reference T_{avg} at rated thermal power, °F |
| P | Measured pressurizer pressure, psig |
| P' | Nominal RCS operating pressure, psig |
| ΔT | Measured ΔT , °F |
| $f_1(\Delta I)$ | = [*] {[*] - ($q_t - q_b$)} when ($q_t - q_b$) < [*] RTP = 0.0 of RTP when [*] RTP < ($q_t - q_b$) < [*] RTP = [*] {($q_t - q_b$) - [*]} when $q_t - q_b > [*]$ RTP Where q_t and q_b are fraction RTP in the upper and lower halves of the core, respectively, and $q_t + q_b$ is the total THERMAL POWER in fraction RTP. |

These values denoted with [] are specified in the COLR.

Overpower ΔT

$$\Delta T \left(\frac{1}{1 + \tau_4 s} \right) \leq \Delta T_0 \{ K_4 - K_5 \left(\frac{1}{(1 + \tau_5 s)} \frac{\tau_3 s T}{(1 + \tau_3 s)} \right) - K_6 [T - T'] - f_2(\Delta I) \}$$

| Parameter | Setpoint |
|--------------|---|
| K_4 | 1.074 |
| K_5 | 0.0188/°F |
| K_6 | 0.0015/°F |
| τ_3 | 10 sec |
| τ_4 | 4 sec |
| τ_5 | 4 sec |
| ΔT_0 | Indicated ΔT at rated thermal power, °F |

T' Reference T_{avg} at rated thermal power, °F
 ΔT Measured ΔT , °F
 $f_2(\Delta I)$ = $[*] \{[*] - (q_t - q_b)\}$ when $(q_t - q_b) < [*] RTP$
= 0.0 of RTP when $[*] RTP < (q_t - q_b) < [*] RTP$
= $[*] \{(q_t - q_b) - [*]\}$ when $q_t - q_b > [*] RTP$
Where q_t and q_b are fraction RTP in the upper and lower halves of the core, respectively, and $q_t + q_b$ is the total THERMAL POWER in fraction RTP.
These values denoted with [] are specified in the COLR.

| | |
|---|--|
| High-pressurizer pressure reactor trip: | 2363 psig |
| Low-pressurizer pressure reactor trip: | 1928 psig |
| Lead time constant: | 9 seconds |
| Lag time constant: | 1 second |
| Low-pressurizer pressure SI: | 1833 psig |
| High-steamline flow SI: | 40 percent flow from 0-20 percent load linearly increasing to 110 percent flow at 100 percent load |
| Low steamline pressure: | 565.3 psig |
| Low T_{avg} : | 542°F |

These assumptions were used as inputs for the analyses in the following subsections. These subsections describe in greater detail each of the transients analyzed.

4.3.1.3 Fifty-Percent Load Rejection from Full-Power Transient

4.3.1.3.1 Description of Analysis and Evaluations

A 50-percent load rejection with steam dump transient was analyzed using the IP2 model of the LOFTRAN code (Reference 1). Since the 50-percent load rejection transient is loop-symmetric, a single-loop version of the LOFTRAN code was used. This computer code is a system-level program code and models the overall NSSS, including the detailed modeling of the control and protection systems.

The 50-percent load rejection is the most severe operational transient that the plant would normally undergo without a reactor trip. The transient was modeled as a turbine runback from 100- to 50-percent power, at a maximum rate of 200-percent per minute. The 200-percent/minute transient is the fastest unloading rate that the turbine can normally perform, so this was used in the analyses.

The RCS average temperature, RCS and pressurizer pressure, and secondary side steam pressure increased rapidly following this transient initiation. The steam dump was available to the condenser, preventing both reactor trip and steam generator safety valve actuation. All NSSS control systems were available to mitigate this transient.

4.3.1.3.2 Acceptance Criteria

The 50-percent load rejection from full-power should provide adequate margins to the nominal trip setpoints (see subsection 4.3.1.2). The plant response should be stable and non-oscillatory. There should be adequate pressurizer PORV capacity to prevent the transient from reaching the high-pressurizer pressure reactor trip setpoint.

4.3.1.3.3 Results

The initial analyses were performed for the low T_{avg} range of operation as noted in subsection 4.3.1.2. While, the results showed margin was needed for the overtemperature ΔT (OT ΔT) trip setpoint (limiting protection system function), at 549°F, as the full-power T_{avg} is increased to the range expected for future SPU operations, larger load rejections can be successfully handled without resulting in a reactor trip,

As the full-power T_{avg} value is increased, the load rejection transient becomes less limiting. This is due to a combination of reasons:

- Higher values of T_{avg} result in more of an initial temperature error to the steam dump control logic, thereby increasing the initial steam dump opening.
- Higher values of T_{avg} result in higher steam pressures, thereby increasing the steam dump flow for a given steam dump valve position.
- Higher values of T_{avg} result in a more negative value of the fuel moderator temperature coefficient (MTC), thereby producing greater fuel reactivity effects to mitigate the transient.

Therefore, as the full-power T_{avg} is increased, larger load rejections can be successfully handled without resulting in a reactor trip. The analyses results indicated that, for full-power T_{avg} values of 558°F and above, the 50-percent design basis load rejection could be accommodated. Therefore, for the full-power T_{avg} value of 562°F at which the plant will operate with the SPU Program implementation, a 50-percent load rejection can be accommodated.

The control system response was smooth during the transient with no oscillatory response noted. All parameters responded smoothly with no sustained or divergent oscillations. This response bounds the higher T_{avg} cases with larger steam dump controller proportional bands and the lower T_{avg} cases.

The peak-pressurizer pressure was controlled by the pressurizer PORV actuation, thereby preventing the pressurizer pressure from reaching the high-pressurizer pressure reactor trip setpoint and showing acceptable capacity for the pressurizer PORVs. The peak steam pressure was no higher than the no-load steam pressure, so the steam generator ARVs were not challenged.

In summary, the plant response for the load rejection transient is acceptable for the SPU program. A 50-percent load rejection (the design basis load rejection) can be accommodated for the full-power T_{avg} value of 562°F, at which the plant will operate with the SPU Program implementation.

4.3.1.4 10-Percent Step-Load Decrease from Full-Power Transient

4.3.1.4.1 Description of Analysis and Evaluations

A 10-percent step-load decrease from full-power transient was analyzed using the IP2 model of the LOFTRAN code (Reference 1). Since the 10-percent step-load decrease transient is loop-symmetric, a single-loop version of the LOFTRAN code was used. This computer code is a system-level program code and models the overall NSSS, including the detailed modeling of the control and protection systems.

The 10-percent step-load decrease was initiated from 100-percent power. Secondary side steam pressure and temperature initially increased, lagged by primary side average temperature (T_{avg}) and pressure increases. The power mismatch between the turbine load and nuclear power, and the resultant temperature error between the T_{avg} and reference temperature (T_{ref}) caused the rods to move into the core, reducing core power. Reactor coolant temperature and pressure were then restored to their equilibrium values.

This transient should not result in the pressurizer pressure reaching the pressurizer PORV actuation setpoint. Stability of the Rod Control System was also assessed.

4.3.1.4.2 Acceptance Criteria

During the 10-percent step-load decrease transient, the PORV actuation setpoint should not be challenged. Therefore, the maximum pressure reached during this transient should be below the PORV actuation setpoint of 2350 psia (2335 psig).

4.3.1.4.3 Results

This transient is the same one that was used to verify acceptability of the pressurizer spray capacity in subsection 4.3.2.2 in this document. The analyses performed for the spray capacity included additional conservatisms not normally used in the plant operability analyses (that is, T_{avg} uncertainty of 7.5°F), and therefore bracketed the best-estimate analyses normally used in the plant operability analyses. The results indicated that no reactor trip setpoints were challenged and the control system response was stable and non-oscillatory. Pressurizer pressure reached a maximum of 2332 psia (2317 psig), for the high T_{avg} case, and the PORVs were not challenged. Therefore, the plant response for the 10-percent step-load decrease transient is acceptable for the SPU Program.

4.3.1.5 10-Percent Step-Load Increase from 90-Percent Power Transient

4.3.1.5.1 Description of Analysis and Evaluations

A 10-percent step-load increase from 90-percent power transient was analyzed using the IP2 model of the LOFTRAN code (Reference 1). Since the 10-percent step-load increase transient is loop-symmetric, a single-loop version of the LOFTRAN code was used. This computer code is a system-level program code and models the overall NSSS, including the detailed modeling of the control and protection systems.

The 10-percent step-load increase was initiated from 90-percent power. Secondary steam pressure and temperature decreased initially, followed by a decrease in the primary side T_{avg} and pressurizer pressure. Pressurizer heaters are actuated to restore system pressure. Normally, the power mismatch between the turbine load and nuclear power, and the resultant temperature error between T_{avg} and T_{ref} would cause the rods to move out of the core, increasing core power till the final 100-percent power condition is reached. However, IP2 has defeated auto rod withdrawal. Therefore, automatic rod control was not credited in the 10-percent step-load increase transient; credit was taken for operator action to withdraw control rods in order to increase reactor power in response to the turbine load increase.

Since the 10-percent step-load increase transient will result in the lowest steam pressure of any of the operational transients, it is analyzed in order to demonstrate that ESF actuation will not occur on low-steam pressure.

4.3.1.5.2 Acceptance Criteria

The 10-percent step-load increase was analyzed to demonstrate that ESF actuation would not occur due to the plant cooldown. The critical function is the ESF actuation on high steamline flow coincident with low steamline pressure (565.3 psig or 580 psia) or low T_{avg} (542°F). While the transient will not actuate the high steamline flow trip setpoint of 110 percent of rated steamline flow at 100-percent power, partial actuation of the other functions could occur. Analyses were performed at the lower range of T_{avg} since this operating condition has the lowest margin to the low steamline pressure or low T_{avg} setpoints. The limiting case is for the minimum full-power steam pressure of 650 psia for the 0-percent tube plugging case (results in minimum full-power T_{avg} of 550.5°F).

4.3.1.5.3 Results

A 10-percent step-load increase transient is not a transient that a plant is expected to experience without planning. Normally plants are operated in a base-loaded fashion at a high power level (at or near 100-percent power). Any power/turbine load increases are known and planned in advance, therefore an operator would be actuating the plant load increase. It would not be an unanticipated transient. While performing the load increase, an operator would also be available to manually withdraw control rods in response to the load increase and to stabilize the plant at the end of the load increase. For the limiting low T_{avg} operating condition (full-power steam pressure of 650 psia, full-power T_{avg} of 550.5°F), the RCS cooldown was only about 3°F below the full-power T_{avg} value and was about 5°F above the low T_{avg} setpoint of 542°F portion of the high steamline flow ESF function. The minimum steam pressure is 630 psia, which is above the low steam pressure setpoint portion of the high steamline flow ESF function (580 psia). The pressurizer level remained well above the low level heater cutoff setpoint of 18 percent of span.

With operator action credited to withdraw the control rods in response to the turbine loading, the transient is mitigated and the 10-percent step-load increase transient can be accommodated without challenging any reactor protection functions.

4.3.1.6 Turbine Trip without Reactor Trip from P-8 Setpoint or Below

4.3.1.6.1 Description of Analysis and Evaluations

A turbine trip without reactor trip transient from the P-8 setpoint or below was analyzed using the IP2 model of the LOFTRAN code (Reference 1). Since the turbine trip transient is loop-symmetric, a single-loop version of the LOFTRAN code was used. This computer code is a system-level program code and models the overall NSSS, including the detailed modeling of the control and protection systems.

The turbine and reactor trip logic was coupled with the P-8 permissive. If a turbine trip occurs from a power level above the P-8 permissive, the turbine trip would actuate a reactor trip. If a turbine trip occurs from a power level at or below the P-8 permissive, no immediate reactor trip would occur. Analyses were performed for the plant P-8 setpoint of 20-percent power. Therefore, a turbine trip without reactor trip transient (that is, turbine trip from power level at or below the P-8 setpoint) can be considered as being a load rejection, and the 50-percent load rejection analyses described in subsection 4.3.1.3 of this report would cover this transient. However, another acceptability requirement of this transient is that the pressurizer PORVs not be actuated. This requirement is the limiting requirement for transient acceptability.

4.3.1.6.2 Acceptance Criteria

The turbine trip without reactor trip transient from the P-8 setpoint or lower power level should provide adequate margins to the nominal trip setpoints (see subsection 4.3.1.2). The plant response should be stable and non-oscillatory. The pressurizer PORVs should not be actuated during this transient. While not a requirement, it is desirable that the steam generator atmospheric relief valves (ARVs) are not challenged during this transient.

4.3.1.6.3 Results

The following assumptions were made besides those described in subsection 4.3.1.2.

- The Rod Control System was assumed to be in manual; no credit was taken for rod motion.

- The analyses were performed for both the 0-percent tube plugging (full-power $T_{avg} = 550.5^{\circ}\text{F}$) and 10-percent tube plugging (full-power $T_{avg} = 559.5^{\circ}\text{F}$) cases for the minimum acceptable full-power steam pressure of 650 psia. Normally, the higher tube plugging case is limiting, but the lower tube plugging case would have the lower ($T_{avg} - T_{no\ load}$) signal to the steam dump valves and, therefore, the greater amount of plant heatup (and resulting higher pressurizer insurge and peak pressurizer pressure). Analyses for these low extremes of full-power T_{avg} would bound the results for higher values of T_{avg} .

The analyses from 20-percent power showed acceptable results for both the 0-percent tube plugging and the 10-percent tube plugging case: the peak pressurizer pressure was 2311 psia for the 0-percent tube plugging case and 2301 psia for the 10-percent tube plugging case. This is an acceptable margin to the pressurizer PORV actuation setpoint of 2350 psia (2335 psig).

The above analyses were performed at the lower limiting T_{avg} values for plant operation at the minimum acceptable full-power steam pressure of 650 psia. As the full-power T_{avg} (and consequentially the full-power steam pressure) was raised above this lower limit, the peak pressurizer pressure was reduced. Therefore, a turbine trip without reactor trip transient is acceptable with the P-8 setpoint set to 20-percent power. A P-8 setpoint of 20-percent power is acceptable for the full-power T_{avg} value of 562°F , at which the plant will operate for the SPU Program implementation.

4.3.1.7 Conclusions

The following was concluded from the plant operability analyses performed:

The pressurizer PORVs would not be challenged for the 10-percent step-load decrease transient.

If operator action is credited for manually withdrawing rods during the 10-percent step-load increase transient, the transient can be accommodated successfully without challenging any reactor trip or Engineered Safety Feature Actuation System (ESFAS) setpoints.

For full-power T_{avg} values of 558°F and above, the 50-percent design basis load rejection can be accommodated. Therefore, the design basis 50-percent load can be accommodated for the full-power T_{avg} value of 562°F , at which the plant will operate for the SPU Program implementation.

The turbine-trip-without-reactor trip from a power level corresponding to the P-8 setpoint or lower can successfully be accommodated for a P-8 setpoint of 20-percent power. With this

value for the P-8 setpoint, the pressurizer PORVs would not be challenged on a turbine-trip-without-reactor-trip transient. A P-8 setpoint of 20-percent power is acceptable for the full-power T_{avg} value of 562°F, at which the plant will operate for the SPU Program implementation.

The control systems are stable and support the SPU Program for all normal condition transients; no long-term, continuous, or diverging plant parameter oscillations were noted during any of the operational transients.

A Pressurizer Level Technical Specification change based on the revised NSSS operating conditions presented in Section 2 will be implemented for the SPU to ensure that the transient increase in pressurizer level will not result in the pressurizer going water solid for any UFSAR transient.

4.3.2 Pressurizer Pressure Control System Component Sizing

The various NSSS pressure control components are intended to maintain the pressurizer pressure at the nominal setpoint during steady-state operation, and to control the pressure excursions that occur during design basis transients to an extent that a reactor trip, ESFAS actuation, or a pressurizer safety valve actuation would not occur. This assessment shows that the installed capacity of the various pressure control components remains acceptable for the SPU conditions.

The following pressure control components were evaluated:

- Pressurizer heaters
- Pressurizer spray valves
- Pressurizer PORVs

Pressurizer Heaters

The pressurizer heaters are sized to be able to heat up the pressurizer liquid at a 200°F/hr rate during the initial plant heatup phase from cold shutdown. In addition, they are intended to assist the plant in controlling the pressurizer pressure decrease that would occur during design basis transients that result in pressurizer outsurge events. These include the initial part of a 10-percent step-load increase transient, a 5-percent per minute plant unloading transient, or events resulting in a reactor trip. The design basis pressurizer heater capacity is 1 kW of heater capacity per cubic foot of pressurizer free volume. Generic analyses on Westinghouse plants have shown that the pressurizer heater capacity is not a strong influence on the minimum pressure noted during the above operational events or during reactor trips. The minimum

pressure is controlled by the outsurge that results during the transient. Analyses have been performed in which the pressurizer heater capacity has been reduced by as much as 20 percent, and no major difference has been observed in the analysis results. The heatup time from cold shutdown to hot standby was not affected by the SPU. The heatup maneuver would be essentially the same as that which IP2 presently experiences. Therefore, the installed pressurizer heater capacity is acceptable for the SPU.

Pressurizer Spray

The design basis for the pressurizer spray capacity is that it is able to handle a 10-percent step-load decrease transient without resulting in the pressure increasing to the pressurizer PORV setpoint. The limiting case is a 10-percent step-load decrease from 100- to 90-percent power.

The SPU power rating would tend to increase the demand on the pressurizer spray. Therefore, the pressurizer spray sizing was analyzed to ensure acceptability. The analysis included the following assumptions:

- The plant is initially at 102 percent of the 3216-MWt SPU power level.
- The plant is initially at nominal $T_{avg} + 7.5^{\circ}\text{F}$ uncertainty.
- The transient is a step-load reduction from the noted 102-percent turbine load to 90-percent load.
- The steam generator heat transfer coefficient increases to the maximum credible value (0-percent fouling, 0-percent SGTP).
- The fuel reactivities are at conservative BOL conditions.
- Credit is taken for automatic operation of all normally functioning NSSS control systems (reactor control, pressurizer pressure and level control, and feedwater control; steam dump is not credited for a 10-percent step-load transient).
- The installed spray capacity analyzed is 325 gpm/valve for a total of 650 gpm.

The limiting case is for the plant operating at the upper limit T_{avg} of 572°F. For this case, the peak pressurizer pressure was 2332 psia, which is below the pressurizer PORV setpoint of 2350 psia. Therefore, the installed pressurizer spray capacity is adequate for the SPU conditions.

Pressurizer PORVs

The design basis for the pressurizer PORV capacity is that it be able to handle a 50-percent load rejection at a maximum turbine unloading rate of 200-percent per minute without resulting in the pressure increasing to the high-pressurizer pressure reactor trip setpoint. The limiting case is a 50-percent load rejection from 100- to 50-percent power.

The higher power rating would tend to increase the demand on the pressurizer PORVs. Therefore, the pressurizer PORV sizing was analyzed to ensure acceptability. The analysis included the following assumptions:

- The plant is initially at 102 percent of the 3216-MWt SPU power level.
- The plant is initially at nominal $T_{avg} + 7.5^{\circ}\text{F}$ uncertainty.
- The transient is a load reduction from the noted 102-percent turbine load to 50-percent load at a maximum rate of 220-percent per minute (200-percent per minute nominal maximum rate, plus 10-percent conservatism).
- The steam generator heat transfer coefficient increases to the maximum credible value (0-percent fouling, 0-percent SGTP).
- The fuel reactivities are at conservative BOL conditions.
- Credit is taken for automatic operation of all NSSS control systems (reactor control, pressurizer pressure and level control, feedwater control, and steam dump control).
- The installed PORV capacity analyzed is 179,000 lb/hr per PORV.

The limiting case for this sizing analysis occurs for the plant operating at the upper limit T_{avg} of 572°F. For this case, the pressurizer PORVs had sufficient capacity to avoid the pressurizer pressure from rising to the implemented high-pressurizer pressure reactor trip setpoint of 2377 psia.

An analysis was also performed for the limiting low T_{avg} condition of a T_{avg} of 550.4°F, which is the minimum full-power T_{avg} that will prevent violation of the steam generator primary-to-secondary pressure limit of 1700 psia. The analysis results showed that the pressurizer PORVs had sufficient capacity to avoid the pressurizer pressure rising to the implemented high-pressurizer pressure reactor trip setpoint of 2377 psia.

Conclusions

Based on this review, the existing pressurizer pressure control component sizing (pressurizer heaters, spray, and PORVs) is acceptable for the SPU conditions.

4.3.3 Overpressure Protection System

As a result of the IP2 SPU Project, the plant operating parameters have changed from the present licensed parameters. The affected parameters are shown in Table 2.1-2. These are at-power parameters. However, the Overpressure Protection System (OPS) only comes into operation during zero-power operation during plant heatup, cooldown, or any operation between cold shutdown and hot standby.

The OPS setpoints would only be required to be evaluated and potentially revised for reasons such as:

- Changes in the design basis transients for which the OPS provides protection against (that is, changes in the design basis mass input or heat input transients). There are no changes in the design basis transients.
- Appendix G pressure-temperature (P-T) limit changes in the adverse direction. Note that a change in the effective full-power years (EFPYs) applicable to the P-T limits does not constitute a reason to revise the setpoints; only an adverse change in the P-T limits themselves would warrant a setpoint re-analysis. There are no changes in the P-T limits.
- Some physical component in the plant changes that affects the performance of the OPS (for example, steam generator replacement, different pressurizer PORV stroke time or flow characteristic, different charging, or SI pump with a revised head/flow curve). The one analysis difference is in the design value of the SGTP level, which is being revised to 10 percent for the SPU Project (see Table 2.1-2 of this report) versus the present 25-percent tube plugging level (see Table 2.1-1 in Section 2 of this report). Therefore, the existing analyses for the 0- to 25-percent tube plugging level bracket the SPU Project 0- to 10-percent plugging level.

Based on this review, the installed OPS setpoints are not affected by the SPU Project.

4.3.4 IP2 SPU Instrumentation and Control Systems

4.3.4.1 Introduction

The Reactor Trip System (RTS), ESFAS, and NSSS Auxiliary System instrumentation have been reviewed to identify changes to setpoints, time constants, logic matrices, electrical power requirements, hardware, separation requirements, and cable routing.

4.3.4.2 I&C Instrumentation Hardware Change

The RTS and ESFAS were reviewed for hardware changes.

The following NSSS Auxiliary Systems were reviewed for hardware changes:

- Chemical and Volume Control System (CVCS)
- Residual Heat Removal System (RHRS)
- Safety Injection System (SIS)
- Containment Spray System (CSS)
- Component Cooling Water System (CCWS)
- Service Water System (SWS)
- Spent Fuel Pit Cooling System (SFPCS)
- Emergency Diesel Generator (EDG) Loading System

4.3.4.3 Equipment Environmental Qualification

Environmental qualification (EQ) (temperature, pressure, humidity) issues for safety-related equipment are addressed in Section 10.8 of this report.

4.3.4.4 Equipment Seismic Qualification

There is no credible reason that the SPU would adversely affect the seismic qualification of existing safety-related equipment. Therefore, the seismic qualification documentation for the existing Westinghouse safety-related equipment is not changed due to the SPU.

4.3.4.5 Instrumentation Settings and Setpoint Changes

The following settings and setpoint changes are due to the SPU:

- Low T_{avg} setpoint

- “K constants” (values for the overtemperature ΔT /overpower ΔT [OT ΔT /OP ΔT] setpoint equations)
- Steam flow transmitters
- Steam flow channel including change to the high steam flow bistable reset value
- Turbine pressure and feed flow channels
- Turbine pressure transmitters
- Low steamline pressure nominal trip setpoint
- Low-pressurizer pressure trip lead/lag values

The safety functions associated with the above changes are not adversely affected.

4.3.4.6 Conclusions

The SPU will require changes to some NSSS instruments and control systems setpoints, time constants, and hardware. However, logic matrices, separation requirements, cable routing, electrical power requirements, and the system safety functions are not required to be changed as a result of the SPU. The setpoint/scaling and time constant changes associated with the SPU are within the capability of the instrumentation. Environmental and seismic issues have been addressed in Section 10.8 of this document. Implementation of the identified changes (hardware, setpoints, re-span, re-calibrate, etc.) configures the instruments and control systems to support the SPU operation. The instrument and control system instrumentation changes have been shown to be acceptable for the SPU.

4.3.5 References

1. WCAP-7878, *LOFTRAN Code Description*, Rev. 6, G. E. Heberle, February 2003.

5.0 NUCLEAR STEAM SUPPLY SYSTEM COMPONENTS

Evaluations were performed to determine the effects of the Indian Point Unit 2 (IP2) Stretch Power Uprate (SPU) parameters on the Nuclear Steam Supply System (NSSS) components. In general, the uprate-related input used for these evaluations are the NSSS parameters (Section 2) and the NSSS design transient changes (Section 3.1). Additional input parameters specific to particular components (for example, NSSS auxiliary equipment design transients for the auxiliary equipment evaluations) were considered and are discussed in the appropriate component evaluation section. The purpose of the evaluations performed for the NSSS components was to confirm that they continue to satisfy the applicable codes, standards, and regulatory requirements under the SPU conditions.

Evaluations were performed in the following areas, and are described within the remainder of this section:

- Reactor vessel structural integrity
- Reactor Pressure Vessel (RPV) System
- Control rod drive mechanisms (CRDMs)
- Reactor coolant loop (RCL) piping and supports
- Reactor coolant pumps (RCPs) and motors
- Steam generators
- Pressurizer
- NSSS auxiliary equipment
- Fracture integrity of NSSS components
- Additional materials considerations for the Reactor Coolant System (RCS)

5.1 Reactor Vessel

5.1.1 Reactor Vessel Structural Integrity

5.1.1.1 Introduction

Evaluations were performed for the Indian Point Unit 2 (IP2) reactor vessel to determine the stress and fatigue usage effects of Nuclear Steam Supply System (NSSS) operation at the revised operating conditions for the Stretch Power Uprate (SPU) Program.

5.1.1.2 Input Parameters and Description of Evaluation Performed

The reactor vessel structural evaluation assesses the effects of the revised operating parameters (Table 2.1-2) and RCS transients on the most limiting locations with regard to ranges of stress intensity and fatigue usage factors in each of the regions as identified in the reactor vessel stress report and addenda. Prior to this evaluation, the most recent vessel structural evaluation for IP2 was performed for the Steam Generator Replacement (SGR) Program. Since the previous design transients (for the SGR Program) for IP2 remain valid and bounding for the SPU Program, no design transient revisions are required. However, the T_{cold} temperature change during the low T_{avg} plant loading and plant unloading at 5 percent per minute increases by 1.5°F as a result of the SPU. Therefore, low T_{avg} plant loading and plant unloading at 5 percent per minute design transients were considered in the reactor vessel evaluation for the SPU Program.

In addition to the plant loading and plant unloading transient revision, the evaluation also considered additional occurrences of the hydro-static test at 2500 psia for the reactor vessel. This was done to supplement the original stress report, which only considered 5 occurrences of hydro-static tests to ASME Section XI pressure test requirements subsequent to commercial operation. These pressure tests are known to occur more frequently than once every 8 to 10 years; therefore, the evaluation considered at least 200 occurrences of the hydro-static test in the maximum cumulative usage factor (CUF) calculation for each reactor vessel region.

Finally, the revised reactor vessel/internals interface loads developed for the SPU Program were evaluated.

5.1.1.3 Acceptance Criteria and Results of Evaluations

The acceptance criteria applicable to the evaluation are as follows:

- The maximum range of stress intensity must be less than 3 times the design stress intensity ($3S_m$) for each location.
- The cumulative fatigue usage factor must be less than unity ($CUF < 1$) for each location.

The evaluation demonstrated that none of the maximum ranges of stress intensity are affected by the plant loading and plant unloading transient revisions. For some locations, the cumulative fatigue usage factors are unaffected by the additional hydro-static tests. The locations where CUFs increased were the outlet and inlet nozzles and supports, the core support pads, the bottom head-to-shell juncture, and the instrumentation tubes. The stress range and CUF results are summarized in Table 5.1-1.

The interface seismic and loss-of-coolant accident (LOCA) reactor vessel and reactor internals (LOCA RV and RI) loads for the SPU are all less than the corresponding faulted condition loads that have previously been considered in the IP2 reactor vessel stress report. Therefore, the loads are acceptable.

5.1.1.4 Conclusions

The maximum ranges of stress intensity are less than the allowable limit of $3S_m$ for all locations except the control rod drive mechanism (CRDM) housings. Although the maximum range of stress intensity for the CRDM housings exceeds $3S_m$, it was justified by a simplified elastic-plastic analysis. The cumulative fatigue usage factors are less than unity for all locations, and the faulted condition interface loads are less than loads used in previous evaluations. In summary, the limits defined in ASME Section III (References 1 and 2) are satisfied, and the SPU will not compromise the structural integrity of the IP2 reactor vessel.

5.1.2 Reactor Vessel Integrity

Reactor vessel integrity is affected by any changes in plant parameters that affect neutron fluence levels or temperature and pressure transients. The neutron fluence projections resulting from the IP2 SPU Program have been evaluated to determine the potential effect on reactor vessel integrity. Typically, such an evaluation is performed by direct comparison of the neutron fluence projections from the analyses of record to the SPU neutron fluence projections. However, prior to the IP2 SPU Program, Westinghouse revised the current reactor vessel integrity analyses of record for IP2 (Reference 3). These revisions extended the pressure-

temperature (P-T) limit curves and documented the bases for the Pressure-Temperature Limit Report (PTLR). The updated reactor vessel integrity evaluations used neutron fluence projections that correspond to 3216 MWt, which is equal to the power level for the IP2 SPU Program. IP2 has already submitted the revised analyses and associated P-T limit curves and received Nuclear Regulatory Commission (NRC) approval (Reference 4).

The evaluations for the SPU build on the approved analyses in WCAP-15629 (Reference 3). The following evaluations were completed for the SPU Program.

- Assessment of the reactor vessel surveillance capsule removal schedule to confirm that the SPU fluence projections do not change the required number of capsules to be withdrawn from the IP2 reactor vessel.
- Assessment of the P-T limit curves to confirm they are based on vessel fluence projections equal to those for the SPU Program (References 3 and 4).
- Review of the RT_{PTS} values to determine if the effects of the SPU fluence projections resulted in an increase in RT_{PTS} for the beltline materials in the IP2 reactor vessel at 32 effective full-power years (EFPYs), which bounds the end of license (EOL) (Reference 3).
- Review of the upper shelf energy (USE) values at 32 EFPYs, which bound the EOL USE values, for all reactor vessel beltline materials in the IP2 reactor vessel to assess the effect of the SPU fluence (Reference 3).

The calculated fluences used in the SPU evaluation comply with Regulatory Guide (RG) 1.190 (Reference 5). These calculations are performed on a plant-specific basis, consistent with the methodology in RG 1.190. It is noted that the fluences used in the baseline analysis differ slightly from those determined for the SPU Program, but not due to the SPU itself. The most recent set of fluence projections considered the power distributions for cycles 17, 18, and 19, whereas the baseline report did not. These cycles have higher peripheral power than the previous cycles used for fluence projections in the prior analysis. The net result was a slight increase in projected fluence. This increased SPU fluence is the basis for the conclusions provided in the following sections.

5.1.2.1 Surveillance Capsule Withdrawal Schedule

The revised SPU fluence projections have been used in the assessment of the current withdrawal schedule for IP2. A calculation of ΔRT_{NDT} at 32 EFPYs was performed to determine the number of capsules to be withdrawn for IP2. This calculation determined that the maximum

ΔRT_{NDT} using the SPU fluences corresponding to 3216 MWt for IP2 at 32 EFPYs is greater than 200°F. These ΔRT_{NDT} values would require 5 capsules to be withdrawn from IP2 (Reference 6). This is consistent with the current withdrawal schedule.

5.1.2.2 Applicability of Heatup and Cooldown Pressure-Temperature Limit Curves

The IP2 Technical Specifications contain P-T limit curves for 25 EFPYs (also documented in Reference 3). These P-T limit curves were based on fluence values that correspond to the SPU power level of 3216 MWt. Therefore, the existing heatup and cooldown curves for 25 EFPYs are acceptable for the SPU without any necessary changes or reduction in EFPYs. In addition, the slight change in fluence due to the updated power distributions (the SPU fluence) also had no effect on the applicability date of the existing P-T limit curves.

5.1.2.3 Emergency Response Guideline Limits

The current peak inside surface RT_{NDT} value at 32 EFPYs (bounding EOL) was calculated to be 246°F for IP2 (Reference 3). The limiting material for IP2 is the intermediate to lower shell girth weld. This RT_{NDT} value places IP2 in Emergency Response Guideline (ERG) Category II. This categorization remains the same for the SPU fluence values.

5.1.2.4 Pressurized Thermal Shock

All beltline materials are expected to have RT_{PTS} values less than 270°F for plates, forgings, and longitudinal welds, and 300°F for circumferential welds. The pressurized thermal shock (PTS) calculations were performed for IP2 using the latest procedures required by the NRC (Reference 7). Based on the evaluation of PTS, all RT_{PTS} values will remain below the NRC screening criteria values using calculated SPU fluence projections that correspond to a SPU power level of 3216 MW through 32 EFPYs (bounding EOL) for IP2 as shown in Table 5.1-2.

5.1.2.5 Upper Shelf Energy

All beltline materials have a USE greater than 50 ft-lb through the EOL (32 EFPYs) as required by the Code of Federal Regulations (CFR) 10CFR50, Appendix G (Reference 8). The 32 EFPYs (bounding EOL) USE was predicted using the EOL 1/4 thickness (1/4t) SPU fluence projections that correspond to a SPU power level of 3216 MWt. The predicted USE values for IP2 have been determined based on bounding SPU fluence values as shown in Table 5.1-3 and documented in WCAP-15629 (Reference 3).

5.1.2.6 Inlet Temperature

Regulatory Guide 1.99, Revision 2 (Reference 9), which is also the basis for 10CFR50.61 (Reference 7), states that "The procedures are valid for a nominal irradiation temperature of 550°F. Irradiation below 525°F should be considered to produce greater embrittlement, and irradiation above 590°F may be considered to produce less embrittlement." The temperature range of 525°F to 590°F serves as the basis of the equations and tables that are used in all the reactor vessel internal analyses described herein. Therefore, the inlet temperature, which is the temperature to which the reactor vessel is subjected, must be maintained within this range to uphold all existing analyses.

5.1.2.7 Conclusions

The fluence projections used for the SPU Program, while considering actual power distributions incorporated to date, have changed slightly due to updated power distributions versus those already incorporated into the reactor vessel integrity analyses of record from WCAP-15629 (Reference 3) (P-T limit curves, ERG category, PTS, and USE). These increased SPU fluence projections have been evaluated against those used for the WCAP-15629 (Reference 3) work and have been determined to result in acceptable conclusions (as noted above) for the SPU Program on reactor vessel integrity.

5.1.3 References

1. *ASME Boiler and Pressure Vessel Code, Nuclear Vessels*, American Society of Mechanical Engineers, New York, 1965 Edition.
2. *ASME Boiler and Pressure Vessel Code, Nuclear Power Plant Components*, American Society of Mechanical Engineers, New York (Appendix F and Appendix I Tables), 1974 Edition.
3. *WCAP-15629, Indian Point Unit 2 Heatup and Cooldown Limit Curves for Normal Operation and PTLR Support Documentation*, T. J. Laubham, Rev. 1, December 2001.
4. *NRC Amendment No. 224, Entergy Nuclear Indian Point 2, LLC Entergy Nuclear Operations, Inc. Docket No. 50-247 Indian Point Nuclear Generating Unit No. 2 Amendment to Facility Operating License*, J. T. Munday (Office of NRR), February 15, 2002.
5. *Regulatory Guide 1.190, Calculational and Dosimetry Methods for Determining Pressure Vessel Neutron Fluence*.

6. ASTM E185-82, *Annual Book of ASTM Standards*, Section 12, Volume 12.02, "Standard Practice for Conducting Surveillance Tests for Light-Water Cooled Nuclear Power Reactor Vessels."
7. 10CFR50.61, *Fracture Toughness Requirements for Protection Against Pressurized Thermal Shock Events*, *Federal Register*, Volume 60, No. 243, December 19, 1995.
8. 10CFR50, Appendix G, *Fracture Toughness Requirements*.
9. Regulatory Guide 1.99, Revision 2, May 1988, "Radiation Embrittlement of Reactor Vessel Materials."

| Table 5.1-1 | | |
|---|-----------------------------------|---------------------------------|
| Maximum Range of Stress Intensity & Cumulative Fatigue Usage Factor Results | | |
| Location | Maximum Range of Stress Intensity | Cumulative Fatigue Usage Factor |
| CRDM Housings | [|] a,c,e |
| Main Closure Closure Head Flange Vessel Flange Closure Studs | [|] a,c,e |
| Outlet Nozzles and Supports Safe End Nozzle | [|] a,c,e |
| Inlet Nozzles and Supports Safe End Nozzle | [|] a,c,e |
| Vessel Wall Transition | [|] a,c,e |
| Core Support Pads | [|] a,c,e |
| Bottom Head-to-Shell Juncture | [|] a,c,e |
| Instrumentation Tubes | [|] a,c,e |
| Head Adapter Plugs | [|] a,c,e |

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

Table 5.1-2

**RT_{PTS} Calculations for IP2 Beltline Region Materials at 32 EFPY with
(3216-MWt) SPU Fluences**

| Material | Fluence (n/cm², E>1.0 MeV) | FF | CF (°F) | ΔRT_{PTS}⁽¹⁾ (°F) | Margin (°F) | RT_{NDT(U)}⁽²⁾ (°F) | RT_{PTS}⁽³⁾ (°F) |
|--|---|-----------|--------------------|---|------------------------|---|--|
| Intermediate Shell Plate B-2002-1 | 1.296 x 10 ¹⁹ | 1.072 | 144 | 154.4 | 34 | 34 | 222 |
| - Using Surveillance Capsule (S/C) Data | 1.296 x 10 ¹⁹ | 1.072 | 114 | 122.2 | 17 | 34 | 173 |
| Intermediate Shell Plate B-2002-2 | 1.296 x 10 ¹⁹ | 1.072 | 115.1 | 123.4 | 34 | 21 | 178 |
| - Using S/C Data | 1.296 x 10 ¹⁹ | 1.072 | 118.2 | 126.7 | 34 | 21 | 182 |
| Intermediate Shell Plate B-2002-3 | 1.296 x 10 ¹⁹ | 1.072 | 176 | 188.7 | 34 | 21 | 244 |
| - Using S/C Data | 1.296 x 10 ¹⁹ | 1.072 | 181.9 | 195.0 | 17 | 21 | 233 |
| Lower Shell Plate B-2003-1 | 1.296 x 10 ¹⁹ | 1.072 | 152 | 162.9 | 34 | 20 | 217 |
| Lower Shell Plate B-2003-2 | 1.296 x 10 ¹⁹ | 1.072 | 142 | 152.2 | 34 | -20 | 166 |
| Intermediate & Lower Shell Longitudinal Welds (Heat # W5214) | 8.741 x 10 ¹⁸ | 0.962 | 230.2 | 221.45 | 65.5 | -56 | 231 |
| - Using S/C Data | 8.741 x 10 ¹⁸ | 0.962 | 254.7 | 245.0 | 44.0 | -56 | 233 |
| Intermediate to Lower Shell Girth Weld (Heat # 34B009) | 1.296 x 10 ¹⁹ | 1.072 | 220.9 | 236.8 | 65.5 | -56 | 246 |

Notes:

1. $\Delta RT_{PTS} = CF \cdot FF$
2. Initial RT_{NDT} values are measured values
3. $RT_{PTS} = RT_{NDT(U)} + \Delta RT_{PTS} + \text{Margin (°F)}$

CF: Chemistry Factor

FF: Fluence Factor

Table 5.1-3

**Predicted 32 EFPY USE Calculations for all the Beltline Region Materials with Bounding
(3216 MWt) SPU Fluences**

| Material | Weight % of Cu | 1/4T EOL Fluence (10¹⁹ n/cm²) | Unirradiated USE⁽¹⁾ (ft-lb) | Projected USE Decrease (%) | Projected EOL USE (ft-lb) |
|--|---------------------------|--|---|---|--|
| Intermediate Shell Plate B-2002-1 | 0.19 | 0.772 | 70 | 20 | 56 |
| Intermediate Shell Plate B-2002-2 | 0.17 | 0.772 | 73 | 21 | 58 |
| Intermediate Shell Plate B-2002-3 | 0.25 | 0.772 | 74 | 32 | 50.3 |
| Lower Shell Plate B-2003-1 | 0.20 | 0.772 | 71 | 27 | 52 |
| Lower Shell Plate B-2003-2 | 0.19 | 0.772 | 88 | 27 | 61 |
| Intermediate & Lower Shell Longitudinal Welds (Heat # W5214) | 0.21 | 0.521 | 121 | 43 | 69 |
| Intermediate to Lower Shell Girth Weld (Heat # 34B009) | 0.19 | 0.772 | 82 ⁽²⁾ | 32 | 56 |

Notes:

1. These values were obtained from original test reports. Values reported in the NRC Database RVID2 are identical with exception to Intermediate Shell Plates B-2002-1, 2. RVID2 reported the initial USE as 76 and 75. This evaluation conservatively used the lower values of 70 and 73.
2. Value was obtained from the average of three impacts tests (71, 84, 90) at 10°F performed for the original material certification.

5.2 Reactor Pressure Vessel System

Evaluations and analyses were performed to assess the effect on the reactor internals components for a Stretch Power Uprate (SPU) at Indian Point Unit 2 (IP2) to a Nuclear Steam Supply System (NSSS) power level of 3230 MWt (core power of 3216 MWt) for the design life of the plant. The analyses/evaluations were performed with 15 x 15 fuel as described in Section 7 of this document.

5.2.1 Introduction

The Reactor Pressure Vessel (RPV) System consists of the reactor vessel, reactor internals, fuel, and control rod drive mechanisms (CRDMs). The reactor internals support and orient the reactor core fuel assemblies and control rod assemblies, absorb control rod assembly dynamic loads, and transmit these and other loads to the reactor vessel. The reactor vessel internal components support in-core instrumentation and also direct coolant flow through the fuel assemblies (core), to provide adequate cooling flow to the various internals structures. The internals are designed to withstand forces due to structure deadweight, fuel assembly pre-load, control rod assembly dynamic loads, vibratory loads, and earthquake accelerations.

Operating a plant at conditions (power and temperature) other than those considered in the original design requires that the interface between the Reactor Vessel System and the fuel be thoroughly addressed to ensure compatibility and to ensure that the structural integrity of the reactor vessel-internals-fuel system is not adversely affected. In addition, thermal-hydraulic analyses are required to determine plant-specific core-bypass flows, pressure drops, and upper head temperatures to provide input to the loss-of-coolant accident (LOCA) and non-LOCA safety analyses, and to NSSS performance evaluations.

The principal areas affected by changes in system operating conditions are:

- Reactor internals system thermal-hydraulic performance
- Rod control cluster assembly (RCCA) scram performance
- Mechanical system evaluations
- Reactor internals system structural response and integrity
- Bottom-mounted instrumentation (BMI) guide tubes and flux thimbles

The major components and features of the reactor internals system for IP2 are summarized as follows. The lower core support assembly consists of the lower support plate, lower support columns, and lower core plate and core barrel, which support the fuel assemblies on the sides and at the bottom. The radial support system, the head-vessel alignment pins, and special temporary guide studs attached to the vessel guide and align the lower core support assembly

during insertion into the reactor vessel. The hold-down spring rests on top of the flange of the lower core support assembly. The upper core support assembly consists of the upper support plate, upper support columns, and upper core plate, and rests on top of the hold-down spring. The guidance and alignment of the upper core support assembly, during its insertion, are provided by the head-vessel alignment pins, the upper core plate alignment pins in the core barrel assembly, and the special temporary guide studs attached to the vessel. The alignment of the core fuel assemblies is provided through the engagement of the lower core plate fuel pins into the bottom of the fuel assemblies and the upper core plate fuel pins into the top of the fuel assemblies. The vessel upper head compresses the hold-down spring, providing joint preload.

The core barrel, which is part of the lower core support assembly, provides a flow boundary for the reactor coolant. When the primary coolant enters the reactor vessel, it impinges on the side of the core barrel and is directed downward through the annulus formed by the gap between the outside diameter of the core barrel and the inside diameter of the vessel. The flow then enters the lower plenum area between the bottom of the lower support plate and the vessel bottom head and is redirected upward through the core. After passing through the core, the coolant enters the upper core support region and then proceeds radially outward through the reactor vessel outlet nozzles. The perforations in the various components, such as the lower support plate, control and meter the flow through the core.

This section summarizes the work performed to assess the effect on the RPV/internals system of the SPU at IP2.

Input Parameters and Assumptions

The principal input parameters used in the analysis of the reactor internal components and RPV system are the NSSS design parameters developed for the SPU (see Table 2.1-2). For structural analysis evaluations, the NSSS design transients discussed in Section 3 were considered. This evaluation considered a full core of 15 x 15 fuel with intermediate flow mixers (IFMs) and with thimble plugging devices removed.

Operating Parameters

The operating parameters (pressure, temperature, flow, and power level) shown in Table 2.1-2 were used in this evaluation. Also, the design transients discussed in Section 3 were used in this evaluation.

A full core of Westinghouse 15 x 15 fuel with IFMs was used in the analysis.

Description of Analyses and Evaluations

Westinghouse has performed evaluations/analyses to assess the effect of the SPU on the RPV/internals system of IP2. The description of various analyses and evaluations are given in the individual subsections, 5.2.1 through 5.2.5.

Acceptance Criteria

The acceptance criteria are listed in each individual section. However, some of the most important acceptance criteria are grouped together and are as follows:

- The design core bypass flow limit with the thimble plugging devices removed is 6.5 percent of the total vessel flow rate.
- Hydraulic lift forces on the reactor internals must be limited so that the internals remain seated and stable.
- For the structural and fatigue evaluations of the various reactor internal components, the cumulative fatigue usage factors must be less than 1.0 for the most critically stressed members.

5.2.2 Thermal-Hydraulic System Evaluations

5.2.2.1 System Pressure Losses

The principal Reactor Coolant System (RCS) flow route through the RPV System at IP2 begins at the inlet nozzles. At this point, flow turns downward through the reactor vessel and core barrel annulus. After passing through this down-comer region, the flow enters the lower reactor vessel dome region. This region is occupied by the internals energy absorber structure, lower support columns, BMI columns, and supporting tie plates. From this region, flow passes upward through the lower core plate and into the core region. After passing up through the core, the coolant flows into the upper plenum, turns, and exits the reactor vessel through the 4 outlet nozzles. The upper plenum region contains support columns and RCCA guide columns.

A key area in evaluation of core performance is the determination of hydraulic behavior of coolant flow within the reactor internals system, that is, vessel pressure drops, core bypass flows, RPV fluid temperatures, hydraulic lift forces, and baffle joint momentum flux. The pressure loss data are necessary inputs to the LOCA and non-LOCA safety analyses and to overall NSSS performance calculations. The hydraulic forces are considered in the assessment

of the structural integrity of the reactor internals, core clamping loads generated by the internals hold-down spring, and the stresses in the reactor vessel closure studs.

The THRIVE computer code was used to perform this evaluation by solving the mass and energy balances for the reactor internals fluid system. This THRIVE analysis determined the distribution of pressure and flow within the reactor vessel, internals, and the reactor core. Results were obtained with a full core of Westinghouse 15 x 15 fuel with IFM grids, thimble plugs removed, and at RCS conditions, as summarized in Table 2.1-2.

5.2.2.2 Bypass Flow Analysis

Description of Analyses

Bypass flow is the total amount of reactor coolant flow bypassing the core region and was not considered effective in the core heat transfer process. Variations in the size of some of the bypass flow paths, such as gaps at the outlet nozzles and the core cavity, occur during manufacturing or change due to fuel assembly changes. Plant-specific, as-built dimensions were used to demonstrate that the bypass flow limits were not exceeded. Therefore, analyses were performed to estimate core bypass flow values to either show that the design bypass flow limit for the plant will not be exceeded, or to determine a revised design core bypass flow.

The present design core bypass flow limit is 6.5 percent of the total reactor vessel flow with the thimble plugging devices removed. This evaluation shows that the design value of 6.5 percent was maintained at the RCS conditions described in Table 2.1-2. The principal core bypass flow paths are described in the following paragraphs.

Baffle-Barrel Region

The current reactor vessel internals configuration incorporates downward coolant flow in the region between the core barrel and the baffle plates. In this configuration, a portion of the coolant exits the reactor vessel inlet nozzle and flows downward in the annulus between the vessel and core barrel. The downward flow passes over the thermal shield to the lower plenum, turns, and flows up through the core region. A portion of this flow enters the baffle-barrel region, which consists of vertical baffle plates that follow the periphery of the core. These are joined to the core barrel by horizontal former plates spaced along the elevation of the baffle plates. At IP2, all but the top former plates have flow holes machined in them. Between the top 2 former levels there are flow holes in the core barrel. Some flow from the vessel and barrel down-comer is diverted through these flow holes, then travels downward through the lower former levels. Most of this baffle/barrel region flow continues down to the top of the lower core plate. There it

passes under the baffle plates and into the bottom of the core. Some fraction of the baffle/barrel plates leaks between the baffle plates and, therefore, is considered as core bypass flow.

Vessel Head Cooling Spray Nozzles

These nozzles provide flow paths between the reactor vessel and core barrel annulus and the fluid volume in the vessel closure head region above the upper support plate. A fraction of the flow that enters the vessel inlet nozzles and into the vessel and barrel downcomer passes through these nozzles and into the vessel closure head region. These flow paths allow circulation of a small fraction of the cold leg coolant into the upper head region of the reactor vessel.

Core Barrel - Reactor Vessel Outlet Nozzle Gap

At IP2, some of the flow that enters the vessel and barrel downcomer leaks through the gaps between the core barrel outlet nozzles and the reactor vessel outlet nozzles and merges with the vessel outlet nozzle flow. Since the lower reactor internals are designed to be removable from the reactor vessel, a small circumferential gap exists at each of the outlet nozzle locations. While the gap is designed to be very small and closes down somewhat at operating conditions due to the differential coefficient of thermal expansion between the reactor internals and the reactor vessel, there is some amount of flow that leaks directly from the vessel inlet/downcomer region and out through these nozzle gaps.

Fuel Assembly - Baffle Plate Cavity Gap

The baffle plates surround the reactor fuel assemblies or core region. The gap between the peripheral fuel assemblies and the baffle plates is defined as the core cavity region. This gap provides the core bypass flow path between the peripheral fuel assemblies and the core baffle plates.

Fuel Assembly Thimble Tubes

Thimble tubes are used as paths for the insertion and removal of control rods, thimble plugging devices, and various core components such as burnable absorbers. These tubes are physically part of each fuel assembly and flow within them is partially effective in removing core heat. However, such flow was analytically not considered to be effective in heat removal, and was consequentially considered to be part of the core bypass flow.

Bypass Flow Analysis Results

Fuel assembly hydraulic characteristics and system parameters, such as inlet temperature, reactor coolant pressure, and flow were used in conjunction with the THRIVE code to determine the effect of SPU RCS conditions on the total core bypass flow. The calculated core bypass flow value was []^{a,c,e} percent with the thimble plugging devices removed at the RCS conditions of Table 2.1-2. Therefore, the design core bypass flow value of 6.5 percent with thimble plugging devices removed, was confirmed to remain bounding.

5.2.2.3 Hydraulic Lift Forces

An evaluation was performed to estimate hydraulic lift forces on the various reactor internal components for the SPU parameters shown in Table 2.1-2. This was done to show that the reactor internals assembly would remain seated and stable for all conditions. The evaluation concluded that the IP2 reactor internals will remain seated and stable for the SPU RCS conditions.

5.2.2.4 Momentum Flux and Fuel Rod Stability

Baffle jetting can be caused a hydraulically-induced instability or vibration of fuel rods induced by a high velocity jet of water. This jet can be created by high-pressure water being forced through gaps between the baffle plates which surround the core. The baffle jetting phenomenon could lead to fuel cladding damage.

At IP2 with SPU conditions and 15 x 15 fuel, the THRIVE evaluations showed that the momentum flux margins were within the design limits and, therefore, baffle jetting is not predicted for IP2 at SPU conditions.

5.2.2.5 Upper Head Fluid Temperatures

The average temperature of the primary coolant fluid that occupies the reactor vessel closure head volume is an important initial condition for certain dynamic LOCA analyses, therefore, it was necessary to determine the upper head temperature for the changes in the RCS conditions. Determination of upper head temperature was derived from the THRIVE evaluations used to assess the core bypass flow. The THRIVE code models the interaction among the different flow paths into and out of the closure head region. Based on this interaction, it calculated the core bypass flow into the head region and the average head fluid temperature based on the different flow path conditions. The IP2 upper head operates at a temperature close to T_{hot} . For IP2, the upper head region best-estimate mean fluid temperature was calculated to be a maximum of 597.4°F for the RCS conditions provided in Table 2.1-2.

5.2.3 RCCA Scram Performance Evaluation

The RCCAs represent perhaps the most critical interface between the fuel assemblies and the other internal components. It is imperative to show that the SPU RCS conditions will not adversely affect the operation of the RCCAs, either during accident conditions or during normal operation.

The IP2 RCCA drop time performance assessment involved the following steps:

- Obtained actual plant drop time-to-dashpot entry data at no-flow and full-flow conditions for each RCCA location.
- Developed an analytical model of the plant's driveline configuration and system operating conditions corresponding to those measurements. A driveline was considered to be that subset of components affecting RCCA drop time. These components were the fuel, upper core plate, upper and lower guide tubes, upper support plate, reactor closure head penetration, thermal sleeve, CRDM, rod travel housing, and the RCCA/drive rod assembly. The system operating conditions included temperature, pressure, and flow. The analytical model included values for parameters that describe geometry of driveline components, component mechanical interaction relationships, hydraulic resistances of flow paths, RCCA/drive rod assembly weight, and system operating conditions.
- Used a coded algorithm previously developed by Westinghouse, with the analytical model, to correlate the model to the plant measured drop times. This algorithm, titled DROP, has been used for this analysis since the original plant design. The DROP algorithm solves Newton's second law of motion. This law states:

$$\Sigma F = (W/g) \times (dV/dt)$$

where:

ΣF = Sum of various forces acting on the RCCA/drive rod assembly at any time (t)

W = total weight of RCCA/drive rod assembly

g = acceleration due to gravity (32.2 ft/sec²)

V = assembly velocity (ft/sec)

t = drop time after CRDM latch release of drive rod (sec)

The correlation involved adjustment of specific code input parameters:

- Characterized RCCA drop performance from no-flow (0 percent) through full-flow (100 percent) based on zero-flow and full-flow core average drop time measurements, and
 - Isolated and accounted for the effects of variations in driveline mechanical interference drag force under normal conditions, and variations in driveline flows across the core, based on core-maximum drop time measurements at zero-flow and full-flow, respectively.
- Adjusted the model (that is, DROP input parameter values) to account for the new system operating conditions being considered due to SPU. Also, conservatively accounted for:
 - Component geometric design tolerances
 - Hydraulic performance uncertainties (related to fuel assembly hydraulic resistance, guide tube/RCCA wear, and reactor coolant flow rate)
 - Abnormal environmental conditions (particularly seismic events)
 - Assessed the effect of such changes in driveline components and/or primary system operating conditions on the limiting RCCA drop time characteristics used in the plant accident analyses. These limiting characteristics were the most severe drop time-to-dashpot entry and normalized RCCA drop time position-versus-time relationship estimated based on the tolerances, uncertainties, and abnormal environmental conditions identified above.

The analysis determined the effect of the conditions shown in Table 2.1-2 on the limiting RCCA drop time. The maximum estimated RCCA drop time with the seismic allowance was calculated to be 1.8 seconds to the top of dashpot. This value is less than the current Technical Specification limit of 2.4 seconds.

5.2.4 Mechanical System Evaluations

The RCS mechanical response to auxiliary line breaks of a LOCA transient is performed in 3 steps. First the RCS is analyzed for the effects of loads induced by normal operation, which includes thermal, pressure, and deadweight effects. From this analysis, the mechanical forces acting on the RPV, which would result from release of equilibrium forces at the break locations,

are obtained. In the second step, the loop mechanical loads and reactor internals hydraulic forces are simultaneously applied, and the RPV displacements due to the LOCA are calculated. Finally, the structural integrity of the reactor coolant loop (RCL) and component supports to deal with the LOCA are evaluated by applying the calculated reactor vessel displacements to a mathematical model of the RCL (see Section 5.4). Thus, the effects of vessel displacements upon the loop and reactor vessel and internals were evaluated.

5.2.4.1 LOCA and Seismic Loads

The RPV LOCA system mathematical model of IP2 was a three-dimensional (3-D) non-linear finite element model that represented the dynamic characteristics of the reactor vessel and its internals in the 6 geometric degrees of freedom. The model was developed using the WECAN computer code. The WECAN computer code (or predecessor codes) was used for this analysis since the original plant design.

The WECAN computer code, which is used to determine the response of the reactor vessel and its internals, is a general purpose finite element code. In the finite element approach, the structure is divided into a finite number of members or elements. The inertia and stiffness matrices, as well as the force array, are first calculated for each element in the local coordinates. Employing appropriate transformation, the element global matrices and arrays are then computed. Finally, the global element matrices and arrays are assembled into the global structural matrices and arrays, and used for dynamic solution of the differential equation of motion for the structure.

To evaluate the effect of changes in RCS conditions on the dynamic response of the RPV System, LOCA analyses were performed to generate core plate motions and the reactor vessel and internals interface loads. The core plate motions were then used to evaluate the structural integrity of the core. Since application of leak-before-break (LBB) methodology has been licensed for the main coolant loop, consideration of breaks in the main coolant loop was not required for structural evaluations (see subsection 5.4.2). The next limiting breaks considered were the branch line breaks. The hydraulic LOCA forces for the breaks listed below were used in the reactor vessel LOCA analysis:

- Accumulator line (cold leg)
- Pressurizer surge line (hot leg)

Following a postulated LOCA, forces were imposed on the reactor vessel and its internals. These forces resulted from the release of the pressurized primary system coolant and, for auxiliary pipe breaks, from the disturbance of the mechanical equilibrium in the piping system prior to the rupture. The release of pressurized coolant resulted in traveling depressurization waves in the primary system. These depressurization waves were characterized by a wavefront

with low pressure on one side and high pressure on the other. The wavefront translated and reflected throughout the primary system until the system was completely depressurized. The rapid depressurization resulted in transient hydraulic loads on the mechanical equipment of the system.

The LOCA loads applied to the RPV System consisted of: reactor internal hydraulic loads (vertical and horizontal), and RCL mechanical loads. All the loads were calculated individually and combined in a time history manner.

The MULTIFLEX computer code calculated the hydraulic transients within the entire primary coolant system. It considered sub-cooled, transition, and two-phase (saturated) blowdown regimes. The MULTIFLEX program uses the method of characteristics to solve the conservation laws, and assumes one-dimensionality of flow and homogeneity of the liquid-vapor mixture.

The MULTIFLEX code considers a coupled fluid-structure interaction by accounting for the deflection of constraining boundaries, which are represented by separate spring-mass oscillator systems. A beam model of the core support barrel was developed from the structural properties of the core barrel. In this model, the cylindrical barrel was vertically divided into various segments and the pressure/wall motions were projected onto the plane parallel to the inlet nozzle on the loop with the postulated auxiliary line pipe break. Horizontally, the barrel was divided into 10 segments, with each segment consisting of 3 separate walls. The spatial pressure variation at each time step was transformed into 10 horizontal forces, which acted on the 10 mass points of the beam model. Each flexible wall was bounded on either side by a hydraulic flow path. The motion of the flexible walls was determined by solving the global equations of motion for the masses representing the forced vibration of an undamped beam.

The severity of a postulated break in a reactor vessel was related to two factors: the distance from the reactor vessel to the break location and the break opening area. The nature of the reactor vessel decompression following a LOCA, as controlled by the internals structural configuration previously discussed, resulted in larger reactor internal hydraulic forces for pipe breaks in the cold leg than in the hot leg (for breaks of similar area and distance from the RPV). Pipe breaks farther away were less severe because the pressure wave attenuated as it propagated toward the reactor vessel. Therefore, pipe breaks at the reactor vessel inlet nozzle were more severe because of the absence of pressure wave attenuation and the structural configuration of the core. In general, the auxiliary line breaks, like the accumulator line and the pressurizer surge line breaks, were not as severe as the main line breaks such as RPV inlet nozzle or RCP outlet nozzle break.

The results of reactor vessel displacements and the impact forces calculated at vessel and internals interfaces were used to evaluate the structural integrity of the reactor vessel and its internals.

The core plate motions for both breaks were used in the fuel grid analysis to confirm the structural integrity of the fuel.

Seismic Analyses

The non-linear time history seismic analyses of the RPV System included the development of the system finite element model and the synthesized time history accelerations.

Similar to the response during LOCA, the RPV System seismic model included sub-models of the reactor vessel, nozzles, internals, fuel and CRDMs. The WECAN finite element model described for LOCA was modified to include the fluid-structure interaction in the RPV model for the seismic safe shutdown earthquake (SSE) time history evaluations. The WECAN reactor vessel-internals-fuel assembly model incorporated the effects of fluid-structure interaction in the downcomer region via hydro-dynamic mass matrices between 2 concentric cylinders (between the core barrel and reactor vessel). The fluid-structure interaction in the seismic analysis was different from that included in the LOCA analysis. In the LOCA analysis, the fluid-structure interaction was included through the MULTIFLEX code; whereas in the seismic analysis, the fluid-structure interaction in the downcomer region (between the core barrel and reactor vessel) was incorporated through the hydro-dynamic mass matrices. The mass matrices with off-diagonal terms were incorporated between nodes on the core barrel and reactor vessel shell.

For a time history response of the RPV and its internals under seismic excitation, synthesized time history accelerations were required. The synthesized time history accelerations for the RPV System analysis were based on the applicable response spectra. The records of a real earthquake, TAFT, were the basis for the synthesized time history accelerations. The spectral characteristics of the synthesized time history accelerations were similar to the original 'TAFT' earthquake records. The resulting north-south, east-west and vertical acceleration time history accelerations were generated for the SSE events.

The results of the system seismic analysis included time history displacements and impact forces for all the major components. The reactor vessel displacements and the impact forces calculated at vessel and internals interfaces were used to evaluate the structural integrity of the reactor vessel and its internals. The core plate motions were used in the fuel grid analysis to confirm the structural integrity of the fuel.

5.2.4.2 Flow-Induced Vibrations

Flow-induced vibrations (FIVs) of pressurized water reactor (PWR) internals have been studied by Westinghouse for a number of years. The objective of these studies was to show that the structural integrity and reliability of reactor internal components are acceptable for plant operating conditions. These efforts have included in-plant tests, scale-model tests, as well as tests in fabricators' shops and bench tests of components, along with various analytical investigations. The results of these scale-model and in-plant tests indicate that the vibrational behavior of two-, three-, and four-loop plants is essentially similar, and the results obtained from each of the tests complement one another and make possible a better understanding of the FIV phenomena.

Based on the analysis for the IP2 reactor internals, the response due to FIVs was extremely small and well within the allowable levels based on the high-cycle endurance limit for the materials.

5.2.4.3 RCCA Insertion Evaluation

To assess the feasibility of crediting the RCCA insertion during a postulated faulted event, the loads on the guide tubes were calculated. These loads included the dynamic loads derived from the RPV System response, subsection 5.2.3.1, the acoustic loads and the cross flow loads during postulated LOCA events. These loads were combined using the square root sum of the squares (SRSS) method. The postulated LOCA events were the 2 limiting breaks stated above, namely, the pressurizer surge line break and the accumulator line break.

The evaluations showed that the maximum LOCA loads were within the allowable loads that were established for 15 x 15 type guide tubes to ensure that the RCCA scram time would be acceptable. Consequently, the RCCA insertion for the IP2 plant could be credited following a faulted-condition event. The evaluation also showed that the maximum seismic load is within the allowable load for the 15 x 15 guide tubes. Therefore, control rod insertion is also ensured during a faulted seismic event.

5.2.5 Structural Evaluation of Reactor Internal Components

In addition to supporting the core, a secondary function of the reactor vessel internals assembly is to direct coolant flows within the vessel. While directing primary flow through the core, the internals assembly also establishes secondary flow paths for cooling the upper regions of the reactor vessel and the internals structural components. Some of the parameters influencing the mechanical design of the internals lower assembly are the pressure and temperature differentials across its component parts and the flow rate required to remove heat generated

within the structural components due to radiation (for example, gamma heating). The configuration of the internals provides adequate cooling capability. The thermal gradients resulting from gamma heating and core coolant temperature changes are maintained below acceptable limits within and between the various structural components.

Structural evaluations demonstrated that the structural integrity of reactor internal components was not adversely affected either directly by the SPU RCS conditions and transients, or by secondary effects on reactor thermal-hydraulic or structural performance. Heat generated in reactor internal components, along with the various fluid temperature changes, resulted in thermal gradients within and between components. These thermal gradients resulted in thermal stresses and thermal growth, which must be considered in the design and analysis of the various components.

The IP2 reactor internals were designed to meet the intent of Subsection NG of the *ASME Boiler and Pressure Vessel Code*, Section III (Reference 1). A plant-specific stress report on the reactor internals was not required. The structural integrity of the IP2 reactor internals design has been ensured by analyses performed on both generic and plant-specific bases. These analyses were used as the basis for evaluating critical IP2 reactor internal components for SPU RCS conditions and revised design transients.

5.2.5.1 Lower Core Plate

Structural evaluations were performed to demonstrate that the structural integrity of the lower core plate was not adversely affected either by the SPU RCS conditions or by secondary effects on reactor thermal-hydraulic or structural performance. For this lower core plate evaluation, the criteria described in Section III, Subsection NG of the *ASME Code* (Reference 1) were used.

Primarily because of the higher gamma heating rates associated with the SPU conditions, the lower core plate is one of the most critically stressed components in the reactor internals assembly. The conclusion of these evaluations was that the structural integrity of the lower core plate was maintained. The SPU RCS conditions resulted in acceptable margins of safety and fatigue usage factors for all ligaments under all loading conditions.

5.2.5.2 Upper Core Plate Evaluations

The upper core plate positions the upper ends of the fuel assemblies and the lower ends of the control rod guide tubes, thus serving as the transitioning member for the control rods in entry and retraction from the fuel assemblies. It also controls coolant flow exiting the fuel assemblies and serves as a boundary between the core and the exit plenum. The upper core plate is restrained from vertical movement by the upper support columns, which are attached to the

upper support plate assembly. Four equally spaced core plate alignment pins restrain lateral movement.

An evaluation was performed to determine the effect of SPU on the structural integrity of the upper core plate. This evaluation concluded that the upper core plate was structurally adequate for the SPU RCS conditions.

5.2.5.3 Baffle-Barrel Region Components

The IP2 lower internals assembly consists of a core barrel into which baffle plates are installed, supported by interconnecting former plates. A lower core support structure is provided at the bottom of the core barrel and a thermal shield surrounds the core barrel. The components comprising the lower internals assembly are precision-machined. The baffle and former plates are bolted into the core barrel. The reactor vessel internals configuration for IP2 uses downward flow in the barrel-baffle region.

Core Barrel Evaluation

The thermal stresses in the core active region of the core barrel shell are primarily due to temperature gradients through the thickness of the core barrel shell. Evaluations were performed to determine the thermal bending and skin stresses in the core barrel for the SPU RCS conditions. These evaluations indicated that the fatigue usage factor, based on all normal/upset conditions, was well below the allowable value of 1.0. From these conservative results, it was concluded that the core barrel was structurally adequate for the SPU RCS conditions.

Baffle-Barrel Bolt Evaluation

The bolts were evaluated for loads resulting from hydraulic pressure, seismic loads, preload, and thermal conditions. The temperature difference between baffle and barrel produced the dominant loads on the baffle-former bolts. Hydraulic pressure and seismic loads produced the primary stresses, whereas bolt preloading and thermal conditions produced the secondary stresses. The SPU RCS conditions did not affect deadweight or preload forces.

Since these bolts are qualified by test, the evaluation of the revised loads consisted of demonstrating that the loads associated with the SPU RCS conditions were bounded by the loads qualified in the test program. Therefore, it was concluded that the baffle-former and barrel-former bolts were structurally adequate for the SPU RCS conditions.

5.2.5.4 Additional Component Evaluations

A series of assessments were performed on reactor internal components that were not significantly affected by the SPU (and the resulting internal heat generation rates), but were affected by the SPU conditions due to primary loop design transients. These components were:

- Lower core support plate
- Lower support columns
- Upper support columns
- Core barrel outlet nozzle
- Core barrel flange
- Lower radial keys
- Upper support assembly (perforated plate)
- Upper support assembly (skirt)
- Upper support assembly (flange)

The results of these assessments, shown in Table 5.2-1, demonstrated that the above listed critical components were structurally adequate for the SPU RCS conditions and the fatigue usage factors were less than 1.0.

5.2.6 BMI Guide Tubes and Flux Thimbles

The BMI guide tubing is typically designed to ASME Section III, Class 1 (Reference 1), but analyzed to Section III, Class 2 rules of NC-3650 due to the size of the guide tubing. The flux thimble is classified as an instrument tube, so it is beyond the jurisdiction of ASME Code per NA-1130(c) (Reference 1). The flux thimbles are qualified as part of BMI guide tubing. No separate qualification of flux thimbles is needed. The weight of the flux thimble is considered in the qualification of BMI guide tubing.

5.2.6.1 Qualification of BMI Tubing and Flux Thimbles

The qualification of the IP2 BMI guide tubing and flux thimble due to the SPU conditions was evaluated to ensure that the BMI guide tubes met allowables.

There are three areas that need to be considered for the reconciliation of BMI guide tubing qualification. They are:

- Pressure increase during transients

- Temperature increase during transients and new core inlet temperature from the SPU parameters (see Table 2.1-2)
- Reactor vessel bottom dome displacement during a LOCA

The BMI guide tubing is qualified for 2500 psia and 550°F, so if the service temperature or pressure values are different than the qualified values, the stress values in the guide tubing must be re-evaluated. Also the reactor vessel displacement at the bottom dome, if different, must be evaluated to determine the stress in the guide tubing.

The evaluation used inputs described in Sections 2 and 3 of this report for temperatures and design transients. Equations 8, 9, 10, 11, and 9-faulted from ASME Section III paragraph NC-3650 (Reference 1) were re-evaluated for the above three changes.

5.2.7 Conclusions

Analyses/evaluations have been performed to assess the effect of changes due to the SPU. The results of these analyses/evaluations demonstrated:

- The use of the design core bypass flow value of 6.5 percent of the total vessel flow rate with thimble plugging devices removed was confirmed for the SPU RCS conditions.
- The IP2 reactor internals assemblies will remain seated and stable at the SPU RCS conditions.
- The RCCA performance evaluation indicated that the current 2.4-second RCCA drop-time from gripper release of the drive-rod-to-dashpot entry limit was satisfied at the SPU RCS conditions and remained conservatively applicable.
- Baffle plate momentum flux margins of safety due to SPU RCS conditions were relatively unchanged from present conditions for mechanical design flow, and remained acceptable.
- Evaluations indicated that the SPU RCS conditions will not adversely affect the response of reactor internals systems and components due to seismic/LOCA excitations and FIVs.
- Evaluations of the critical reactor internal components indicated that the structural integrity of the reactor internals was maintained at the SPU RCS conditions. Limiting CUFs were all shown to be less than 1.0.

- The stresses in the BMI guide tubing were within the allowables and meet the requirements of ASME Section III, paragraph NC-3650 (Reference 1). The new stress values are compared with their allowables in Table 5.2-2.

5.2.8 References

1. *ASME Boiler and Pressure Vessel Code, Section III, "Rules for Construction of Nuclear Vessels,"* 1965 Edition with Winter 1965 Addenda, The American Society of Mechanical Engineers, New York, NY.

Table 5.2-1

IP2 – SPU

Summary of Critical Reactor Internal Components Stresses and Fatigue Usage Factors

| Component | Normal/Upset Conditions | | | | Faulted Conditions | |
|--|-------------------------|------------------|----------------------|-------------------------|--------------------|------------------|
| | (Pm) psi | (Pm + Pb) psi | (Pm + Pb + Q) psi | Fatigue Usage Factor | (Pm) psi | (Pm + Pb) psi |
| Lower Core Support Plate | | | | | | a,c,e |
| Core Barrel – Lower Girth Weld | | | | | | |
| Lower Support Column | | | | | | |
| Mid Core Barrel | | | | | | |
| Upper Core Barrel | | | | | | |
| Core Barrel Nozzle | | | | | | |
| Core Barrel Flange | | | | | | |
| Lower Radial Key base | | | | | | |
| Lower Radial Key (45° Plane) | | | | | | |
| Upper Support Assembly – Perforated Plate (Center) | | | | | | |
| Upper Support Assembly - Skirt | | | | | | |
| Upper Support Assembly - Flange | | | | | | |

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

| Table 5.2-2 Maximum Stresses for BMI Tubes | | |
|---|--------------|------------------------|
| Equation No. | Stress (psi) | Allowable Stress (psi) |
| 8 |] | a,c,e |
| 9 | | |
| 10 | | |
| 11 | | |
| 9-Faulted | | |

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

5.3 Control Rod Drive Mechanisms

5.3.1 Introduction

This section addresses the ASME Code of Record structural considerations for the pressure boundary components of the Westinghouse full-length L-106 control rod drive mechanisms (CRDMs). The CRDMs were evaluated for the Indian Point Unit 2 (IP2) Stretch Power Uprate (SPU) conditions.

5.3.2 Input Parameters and Assumptions

The Model L-106 CRDMs were originally designed and analyzed to meet the ASME Code 1965 Edition through the Summer 1966 Addenda or later (Reference 1). The Nuclear Steam Supply System (NSSS) design parameters for the IP2 SPU are provided in Table 2.1-2 of this report and the NSSS design transients are discussed in Section 3.1. The seismic loading has not been changed for the IP2 SPU Program.

The IP2 CRDMs operate with a T_{hot} upper head condition, defined by the vessel outlet reactor coolant temperature of the SPU parameters, and must be analyzed for the NSSS design transients defined for the hot leg. The differences associated with the uprating requirements are discussed in subsection 5.3.3 of this report.

5.3.3 Description of Analysis

5.3.3.1 Operating Pressure and Temperature

The Reactor Coolant System (RCS) temperature and pressure values were compared to the current design analysis for the CRDMs. There are no changes from the current reactor coolant pressure of 2250 psia for any of the uprating cases from the SPU parameters for IP2. The hot-leg temperature (T_{hot}) defined by the vessel outlet temperature on the parameters for the IP2 SPU is a maximum of 605.8°F, which is less than the 650.0°F temperature used in the original analysis of record. Since none of the temperatures exceeds the previously analyzed temperature, and the pressure does not change, the SPU parameters are bounded by the current analyses of record.

Table 5.3-1 summarizes the hot-leg parameters. From Table 5.3-1, the SPU conditions provide an RCS T_{hot} of 605.8°F, compared to 611.7°F for the 1990 Uprate Program (Reference 2). Therefore, the 1990 analysis range bounds the range of T_{hot} for the SPU Program.

5.3.3.2 Transient Discussion

The Section 3.1 NSSS design transients have not been changed from the currently analyzed condition for the IP2 CRDMs. These transients were evaluated in detail for the 1990 Uprate Program for IP2 and remain applicable for the SPU Program. There are no changes in the pressure transients associated with these system transients.

5.3.4 Acceptance Criteria

The acceptance criteria for the ASME Code structural analysis of the CRDM pressure boundary are that the analyzed stresses do not exceed the stress allowable of the ASME Code and that the cumulative fatigue usage factors from the code fatigue analysis remain less than 1.0. When the 1990 uprate program evaluations were performed (Reference 1), the stresses and the cumulative usage factors were increased for those cases for which changes to the design transients indicated that an increase was necessary. However, for those cases for which changes to the design transients would have allowed a decrease in stresses or cumulative usage factors, no decrease was calculated, and no credit was taken for such a decrease. Therefore, if the NSSS design transients are shown to be bounded by those considered for either the 1990 Uprate Program or for the original design, then the stresses and cumulative usage factors calculated for the CRDMs for the 1990 Uprate Program remain bounding and applicable for the SPU Program.

5.3.5 Results

A summary of the results of the analysis performed for the 1990 Uprate Program is presented in Tables 5.3-2 and 5.3-3. The highest calculated stresses, as compared to the associated allowables, are presented in Table 5.3-2 for the upper, middle, and lower joints of the CRDM pressure boundary. The cumulative usage factors that were calculated for the 1990 Uprate Program are given in Table 5.3-3. It is noted that the highest cumulative usage factor, at the upper joint canopy, was calculated in a conservative manner where the applied transients were grouped for analysis and the allowable number of cycles considered for each group was based on the most severe transient in the group.

5.3.6 Conclusions

The SPU Program parameters and NSSS design transients are bounded by the parameters and transients considered for either the 1990 Uprate Program or the original design analysis. Therefore, the evaluation results for the SPU Program are consistent with, and continue to comply with, the current licensing basis and acceptance requirements for IP2.

5.3.7 References

1. *ASME Boiler and Pressure Vessel Code, Section III, Rules for Construction of Nuclear Vessels*, 1965 Edition through Summer 1966 Addenda, The American Society of Mechanical Engineers, New York.
2. WCAP-11972, *Consolidated Edison Company of New York, Inc., Indian Point Unit 2, NSSS Stretch Rating – 3083.4 MWt Licensing Report*, Rev. 1, March 1989.

| Table 5.3-1 NSSS Conditions Used to Bracket All Operating Conditions for IP2 SPU | | | | |
|--|------------------------------|----------------------|-----------------------|----------------------|
| Parameter | 1990 Uprate Program Analysis | | 3216 MWt Analysis | |
| | High T _{avg} | Low T _{avg} | High T _{avg} | Low T _{avg} |
| T _{hot} | 611.7°F | 582.2°F | 605.8°F | 583.7°F |

| Table 5.3-2 Highest Stresses Compared to Allowable for CRDM Joints, Applicable for IP2 SPU | | |
|--|--|-----------------|
| CRDM Joint and Component | Stress (psi) | |
| | Values Applicable for the SPU Program | Allowable Value |
| Upper Joint Threaded Area | [| a,c,e |
| Middle Joint Canopy | [|] |
| Lower Joint Canopy | [|] |

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

| Table 5.3-3 Cumulative Usage Factors for CRDM Joints Applicable for IP2 SPU | | |
|---|---------------------------------------|-----------------|
| CRDM Joint and Component | Cumulative Usage Factor | |
| | Values Applicable for the SPU Program | Allowable Value |
| Upper Joint Canopy | [^{a,c,e}] | 1.00 |
| Upper Joint Canopy Weld | [] | 1.00 |
| Upper Joint Threaded Area | [] | 1.00 |
| Middle Joint Canopy Weld | [] | 1.00 |
| Lower Joint Canopy Weld | [] | 1.00 |

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

5.4 Reactor Coolant Loop Piping and Supports

5.4.1 Reactor Coolant Loop Piping

5.4.1.1 Introduction

The parameters associated with the Stretch Power Uprate (SPU) Program were evaluated and analyzed to determine the effects on the existing design basis reactor coolant loop (RCL) analysis for the following components:

- RCL piping stresses and displacements
- Primary equipment nozzle loads
- Pressurizer surge line piping stresses and displacements including the effects of thermal stratification
- RCL branch nozzle loads
- Class 1 and 2 auxiliary piping systems

5.4.1.2 Inputs and Assumptions

The following four basic sets of input parameters were considered in the evaluation:

- Nuclear Steam Supply System (NSSS) design parameters (Table 2.1-1 of Section 2)
- NSSS design transients (Section 3 of this report)
- LOCA hydraulic forcing functions loads (Section 6.7 of this document) and associated reactor pressure vessel (RPV) motions (Section 5.2 of this document)
- Feedwater thrust and jet forces from the previous analyses

The acceptance criteria for the Indian Point Unit 2 (IP2) RCL Piping System as indicated in the *Indian Point Nuclear Generating Unit No. 2 Updated Final Safety Analysis Report* (Reference 1) are based upon the *ANSI Code for Pressure Piping, Power Piping USAS B31.1*, 1955 Edition (Reference 2). In the evaluation performed for the SPU Program, the acceptance criteria are based on the *ANSI Code for Pressure Piping, Power Piping USAS B31.1*, Summer 1973 (Reference 3) to be consistent with the RCL piping analysis for the Replacement Steam

Generator (RSG) and Snubber Reduction Program and documented in the current analysis of record. Therefore, the acceptance criteria for the RCL analysis are based on *USAS B31.1 Power Piping Code*, Summer 1973 Edition (Reference 3), and as specified in the analysis of record.

The acceptance criteria for the pressurizer surge line thermal stratification analysis are those in the *American Society of Mechanical Engineers (ASME) Boiler & Pressure Vessel (B&PV) Code* Section III, Subsection NB, 1986 Edition (Reference 4), as specified in WCAP-12937 (Reference 5).

The parameters associated with the SPU Program were reviewed to determine the effects on the existing RCL piping and the subsequent effects on the RCL branch nozzles and the Class 1 and 2 auxiliary lines attached to the RCL. The conclusions of this review are summarized in subsection 5.4.1.4.

5.4.1.2.1 NSSS Design Parameters

The NSSS design parameters (see Table 2.1-2 of this report) were used in the thermal analysis of the RCL. These parameters were compared to the parameters used in the RSG Program analyses. The RCL was evaluated for two temperature conditions—one for the lower-bound temperature condition (Cases 1 and 2), and the second for the upper-bound temperature condition (Cases 3 and 4) as identified in Table 2.1-2. The lower bound and upper bound hot leg temperatures for the SPU Program were bounded by the hot leg temperatures for the RSG case. The upper bound temperatures for the RSG Case were also bounding for both the crossover legs and the cold legs. The lower bound temperatures for the SPU Program decreased by an insignificant amount from the RSG case, and this variation in temperatures was considered small and insignificant. The design basis parameters summarized in WCAP-12937 (Reference 5) for the pressurizer surge line remain applicable.

5.4.1.2.2 NSSS Design Transients

The effect on design transients due to the changes in full-power operating temperatures for the SPU is addressed in Section 3 of this report. As per Section 3, the current NSSS design transients remain bounding for the SPU and the existing transients remain valid. Additionally, per the current analysis of record, the design criteria for the RCL piping is *USAS B31.1 Power Piping Code*, Summer 1973 Edition (Reference 3); thus, no fatigue analysis is required for the RCL.

For the pressurizer surge line, the effect of the design transients with respect to the thermal stratification and fatigue analysis was controlled by the ΔT between the pressurizer temperature

and the hot-leg temperature. The controlling ΔT s for the pressurizer surge line were associated with heatup and cooldown events that were not affected by the SPU. Therefore, the SPU will have no adverse effect on either the thermal stratification or the fatigue analysis for the pressurizer surge line, and the limiting transients in WCAP-12937 (Reference 5) remain valid.

5.4.1.2.3 LOCA HFFs Functions Loads and Associated RPV Motions

The effect on the LOCA hydraulic forcing functions (HFFs) due to the SPU Program is addressed in Section 6.7 of this report. Leak-before-break (LBB) is applicable for the RCL main loop piping (see subsection 5.4.2). Based on the application of LBB, the RCL was evaluated for LOCA using HFFs generated for the SPU Program, based on breaks at the 14-inch surge line nozzle, at the 14-inch RHR line nozzle on the hot leg, and at the 10-inch accumulator line nozzle on the cold leg. RPV motions corresponding to the surge line break, RHR line break, and accumulator line break were also used.

5.4.1.2.4 MSLB and FLB Thrust and Jet Forces and Associated Steam Generator Motions

The RCL was evaluated for secondary side breaks at the main steam line and feedwater line terminal end nozzle locations at the steam generator.

5.4.1.3 Analysis Methods

The system analysis of the RCL piping was performed for all applicable deadweight, thermal expansion, seismic, and LOCA and pipe break cases.

5.4.1.4 RCL Piping Analysis and Results

The deadweight analysis for the SPU considered the weight of the RCL piping, including the weight of the contents and the primary equipment water weight.

The thermal analysis considered the range of operating temperatures for 100-percent power as defined by the SPU NSSS parameters identified in Table 2.1-2 of this report. Since the temperature ranges of the RSG Program are reconciled to remain applicable to the corresponding temperature ranges for the SPU Program, the shimming performed for the RSG Program was reconciled and determined to be applicable for the SPU Program.

The seismic analysis methods were the same as used in previous analyses. The design basis earthquake (DBE) input response spectra for the SPU parameters were those used in the current analysis of record for the RSG Program. The seismic analyses considered multiple cases based on various primary equipment supports being active with other supports being

inactive, which accounted for the range of operating temperatures as defined by the SPU NSSS temperatures in Table 2.1-2.

The LOCA analysis for the RCL considered time history hydraulic forces distributed throughout the RCL system and included the effects of the associated RPV motion. The analysis was performed for the breaks at the auxiliary nozzles for the 14-inch residual heat removal (RHR) line on the hot leg, the 14-inch surge line nozzle on the hot leg, and the 10-inch accumulator line on the cold leg. IP2 has been licensed for LBB on the main RCL piping.

Secondary side breaks at the main steam line and feedwater line terminal end nozzle locations at the steam generator were included in the analyses and results were shown to be acceptable.

Evaluations were performed for the SPU NSSS design parameters, NSSS design transients, LOCA HFFs and associated RPV motions, and the main steam line break (MSLB) and feedwater line break (FLB) thrust and jet forces. Based on these evaluations, the RCL piping stresses, displacements, and primary equipment nozzle loads were acceptable for the SPU Program.

The maximum RCL piping stresses for the RCL piping and the corresponding code-allowable stress values are presented in Table 5.4-1. The stresses were combined in accordance with the methods specified in the current analysis of record (AOR).

The applicable RCL piping primary equipment support loads for the SPU parameters were provided for evaluation. Confirmation of acceptability is discussed later in subsection 5.4.3.

The primary equipment nozzle loads were compared to the allowables as defined in the equipment design specifications and the loads previously evaluated for the RSG Program. The nozzle loads are acceptable and the SPU Program has no adverse effect on analysis results.

The applicable RCL piping loads resulting from the range of operating temperatures, as defined by the SPU NSSS parameters, were provided for evaluation and confirmation of LBB criteria (see subsection 5.4.2).

The SPU effect on RCL piping displacements at the RCL branch nozzles and corresponding Class 1 and Class 2 auxiliary piping systems was evaluated. This evaluation considered the SPU parameters, SPU LOCA HFFs, and the NSSS fluid system performance evaluation in Section 4 of this report. This evaluation included the RCS, Primary Sampling System (PSS), Chemical and Volume Control System (CVCS), Residual Heat Removal System (RHRS), Safety Injection System (SIS), Component Cooling Water System (CCWS), and the Containment Spray System (CSS). The SPU effect on RCL piping displacements at branch nozzles had a

negligible effect on the RCL branch nozzle loads and on the Class 1 and Class 2 auxiliary piping systems that are attached to the RCL.

Based on evaluation of the NSSS parameters and the design transients for the SPU, the current design basis pressurizer surge line analysis results (Reference 5), including the effects of thermal stratification, are applicable and remain valid for the SPU Program.

5.4.2 Application of LBB Methodology

The current structural design basis of IP2 includes the application of LBB methodology to eliminate consideration of the dynamic effects resulting from pipe breaks in the RCS primary loop piping. This section describes the analyses and evaluations performed to demonstrate that the elimination of these breaks continues to be justified at the operating conditions associated with the IP2 SPU Program.

Introduction

Westinghouse performed analyses for the LBB of IP2 primary loop piping in 1986 and 1989. The results of the analyses were documented in WCAP-10977, Revision 2 (Reference 6) and WCAP-10977 Supplement 1 (Reference 7) and approved by the Nuclear Regulatory Commission (NRC). Westinghouse also performed analyses in 2000 to support the RSG Program and performed an evaluation for the Snubber Reduction Program.

To demonstrate the elimination of RCS primary loop pipe breaks, the following objectives had to be achieved:

- Demonstrate that margin exists between the "critical" crack size and a postulated crack that yields a detectable leak rate.
- Demonstrate that there is sufficient margin between the leakage through a postulated crack and the leak detection capability.
- Demonstrate margin on the applied load.
- Demonstrate that fatigue crack growth is negligible.

These objectives were met by the previous analyses.

To support the IP2 SPU Program, the previous LBB analyses were updated to address SPU conditions. The SPU evaluation and results are addressed below.

Input Parameters and Assumptions

The loadings, operating pressure, and temperature parameters for the SPU were used in the evaluation.

The parameters, which are important in the evaluation, are the piping forces, moments, normal operating temperature, and normal operating pressure. These parameters were used in the evaluation. For normal operating temperature and normal operating pressure at the SPU conditions, see Section 2 of this report.

Description of Analyses and Evaluations

The recommendations and criteria proposed in the *Standard Review Plan (SRP)* (Reference 8) were used in this evaluation. The primary loop piping dead weight, normal thermal expansion, SSE, and pressure loads due to the SPU Program have been used. The normal operating temperature and pressure due to the SPU conditions were used in the evaluation. The evaluation showed that all the LBB recommended margins were satisfied for the SPU conditions. The margins from the SRP (Reference 8) are also described below.

Acceptance Criteria and Results

The LBB acceptance criteria is based on the SRP 3.6.3 (Reference 8). The recommended margins are as follows:

- Margin of 10 on leak rate
- Margin of 2 on flaw size
- Margin on loads of 1 (using faulted load combinations by absolute summation method)

The evaluation results showed the following at all the critical locations:

Leak Rate - A margin of 10 exists between the calculated leak rate from the leakage flaw and the leak detection capability of 1 gpm.

Flaw size - A margin of 2 or more exists between the critical flaw and the flaw having a leak rate of 10 gpm (the leakage flaw).

Loads - A margin of 1 on loads exists.

The evaluation results show that the LBB conclusions provided in WCAP-10977, WCAP-10977 Supplement 1 (References 6 and 7), and analyses performed in 2000 to support the

Replacement Steam Generator (RSG) Program as well as an evaluation for the Snubber Reduction Program for IP2 remain unchanged for SPU.

Conclusions

The LBB acceptance criteria are satisfied for the IP2 primary loop piping at the SPU conditions. All the recommended margins are satisfied and the conclusions shown in WCAP-10977, WCAP-10977 Supplement 1 (References 6 and 7), and analyses performed in 2000 to support the RSG Program as well as an evaluation for the Snubber Reduction Program remain valid. It is, therefore, concluded that the dynamic effects of RCS primary loop pipe breaks need not be considered in the structural design basis of IP2 at the SPU conditions.

5.4.3 RCS Equipment Supports

5.4.3.1 Introduction

This report documents the acceptability of the equipment supports for the SPU conditions. The parameters associated with the SPU Program were reviewed to determine the effects of the SPU conditions on the existing design basis analysis (DBA) for the RCS equipment supports (RCSES). The following loads were considered in the analysis:

- Piping loads on RCSES
 - Deadweight
 - Thermal
 - Operating basis earthquake (OBE) and DBE
 - Pipe break (main steam and feedwater)
 - LOCA (pressurizer surge line, residual heat removal, 45-degree and 90-degree accumulator)
- Loads due to attachments to RCSES
 - Pipe supports
 - Whip restraints
- Pipe whip and jet impingement loads on RCSES
 - From 10-inch lines and larger

5.4.3.2 Inputs

The following sets of inputs were used in the evaluation:

- RCSES drawings and embedment allowables
- Pipe whip and jet impingement loads
- Support attachment loads
- RCL piping loads
- Reactor vessel loads

The IP2 RCL was re-analyzed to incorporate the SPU conditions. The RCL piping analysis calculated revised loads from the piping to the steam generator and RCP supports. The reactor vessel analysis calculated revised loads for the reactor vessel support reconciliation.

5.4.3.3 Analysis Methods

The structural analyses of the RCSES used the structural analysis computer program STAAD/Pro Release 2000 (Reference 9) for all structural modeling.

The RCL equipment support loads result from several different loading conditions:

- **Deadweight** Loads due to deadweight of equipment, attached piping, insulation, and contained fluids
- **Thermal** Load on the supports due to constrained thermal expansion of the RCL
- **OBE** Seismic loads due to the OBE
- **DBE** Seismic loads due to the DBE
- **Main steamline** Loads due to a break in the main steam line break (MSLB)
- **Feedwater line** Loads due to a break in the main feedwater line break (FLB)
- **LOCA** Loads due to a break in any one of several RCL nozzles, that is, surge line nozzle, RHR line nozzle, or accumulator line nozzle

The load combinations were based on Table 1.11-2 of the UFSAR (Reference 1).

The loading combinations were applicable for all support components. However, the allowable stress and loads were different and were addressed separately for the steam generator and RCP frames, the tie rods, snubbers, equipment hold-down bolts, embedments, and the RPV support.

5.4.3.4 Acceptance Criteria

The acceptance criteria for the IP2 RCSES as indicated in the *Indian Point Nuclear Generating Unit No. 2 Updated Final Safety Analysis Report* (Reference 1) are based upon Table 1.11-2, in combination with the criteria discussed below.

Steam Generator and RCP Frames

Per Section 4.1.7 and Table 4.1-9 of the UFSAR, the original piping design code for IP2 supports is the 1955 Edition of USAS B31.1 (Reference 2). The USAS B31.1 Code does not provide detailed guidance for the evaluation of piping supports. Common practice is to use the American Institute of Steel Construction (AISC) Specification (Reference 10).

The Sixth Edition AISC Specification (Reference 10) was used for evaluating the piping supports.

For Load Case 1, the allowable stresses provided in the AISC code were used (allowables were based on the actual temperature of the steel). For Load Case 2, the allowables could be increased by 1/3, however, compressive buckling stresses were limited to 2/3 of critical buckling.

For Load Cases 3 and 4, well-defined criteria were not available in the AISC Specification. The criterion is that "Deflections and stresses of supports limited to maintain supported equipment within their stress limits." This correlates to limiting the deflection of the supports such that additional stresses do not occur in the supported piping/equipment. Acceptable means of satisfying the above criteria are to use the faulted increase factors provided in Appendix F of the 1974 ASME Section II Code for Supports, that is, F-1370(a) and F-1370(c) (Reference 11). These rules state that the increase factor for faulted-condition loads can be increased above the Level A (AISC allowables) by:

- Increase factor = minimum ($1.2 \times S_y / F_t$, $0.7 \times S_u / F_t$), since $F_t = 0.6 F_y$ for the frame members being considered

- Increase factor = minimum {2, $0.7 \times S_u / (0.6 \times S_y)$ }
- Section F-1370(c) states that loads shall not exceed 2/3 of the critical buckling load

Steam Generator and RCP Tie Rods

The steam generator and RCP tie rods are strictly tension members. As such, the allowable loads were based on the minimum of the turnbuckle allowables, the tensile area of the tie rods, and the compressive area under the nuts.

Concrete Embedments

The embedment allowable loads were taken from design information used in previous analyses.

Snubbers

The maximum allowable load per snubber was taken from design information used in previous analyses.

RCP and Steam Generator Holddown Bolts

The RCP bolts evaluation was in accordance with AISC, Seventh Edition (Reference 12). For Load Cases 3 and 4, the guidance provided in ASME Code Case 1644-6 (Reference 13) was used.

The steam generator feet connections were evaluated by comparison with design information used in previous analyses.

RPV Supports

The reactor vessel support evaluations were based on WCAP-9117 (Reference 14). The letter from W. J. Cahill, Jr. to the Director of Nuclear Regulatory Regulation, June 15, 1978, addresses the applicability of WCAP-9117 to Indian Point Unit 2 (Reference 15).

5.4.3.5 RCSES Analysis and Results

The loads on the steam generator, RCP, and RPV supports meet the acceptance criteria provided in subsection 5.4.3.4 of this report.

A summary of the results is provided in Table 5.4-2.

5.4.4 References

1. *Indian Point Nuclear Generating Unit No. 2, Updated Final Safety Analysis Report, Docket No. 50-247.*
2. *United States of America Standards (USAS), B31.1 Power Piping Code, 1955 Edition.*
3. *United States of America Standards (USAS), B31.1 Power Piping Code, Summer 1973 Edition.*
4. *ASME Boiler and Pressure Vessel Code, Section III, Subsection NB, 1986 Edition, The American Society of Mechanical Engineers, New York, NY.*
5. *WCAP-12937, Structural Evaluation of Indian Point Units 2 and 3 Pressurizer Surge Lines, Considering the Effects of Thermal Stratification, May 1991.*
6. *WCAP-10977 (Proprietary), Technical Bases for Eliminating Large Primary Loop Pipe Rupture as the Structural Design Basis for Indian Point Unit 2, Rev. 2, F. J. Witt, et al., December 1986.*
7. *WCAP-10977 Supplement 1, Additional Information in Support of the Technical Justification for Eliminating Large Primary Loop Pipe Rupture as the Structural Design Basis for Indian Point Unit 2, January 1989.*
8. *NRC Standard Review Plan, Public Comment Solicited, 3.6.3, "Leak-Before-Break Evaluation Procedures," Federal Register/Vol. 52, No. 167/Friday, August 28, 1987/Notices, pp. 32626-32633.*
9. *STAAD/Pro STAAD-III Revision 2000 Computer Program.*
10. *American Institute of Steel Construction (AISC) Specification for Design, Fabrication & Erection of Structural Steel for Buildings, Sixth Edition, 1963.*
11. *ASME Boiler and Pressure Vessel Code, Section II, Appendix F, 1974.*
12. *American Institute of Steel Construction (AISC) Specification for Design, Fabrication and Erection of Structural Steel for Buildings, Seventh Edition, 1973.*
13. *ASME Boiler and Pressure Vessel Code, Code Case 1644-6, The American Society of Mechanical Engineers, New York, NY.*

14. *WCAP-9117, Analysis of Reactor Coolant System for Postulated Loss-of-Coolant Accident: Indian Point 3 Nuclear Power Plant, June 1977.*
15. *Letter W. J. Cahill, Jr. to the Director of Nuclear Regulatory Regulation, Applicability of WCAP-9117 to Indian Point Unit 2, June 15, 1978.*

Table 5.4-1

RCL Piping Stress Analysis Summary – SPU Program

| Stress Combination | Hot Leg | | Crossover Leg | | Cold Leg | |
|--|---------------------------------------|------------------|---------------------------------------|------------------|---------------------------------------|------------------|
| | Maximum ksi | Allowable ksi | Maximum ksi | Allowable ksi | Maximum ksi | Allowable ksi |
| Normal Loads Combination (pressure + deadweight) | [| | | | | a,c,e] |
| Allowable Stress Limit | (1.0 S) | | (1.0 S) | | (1.0 S) | |
| * Normal + Maximum Potential Earthquake (DBE) Loads Combination (pressure + deadweight+DBE) | [| | | | | a,c,e] |
| Allowable Stress Limit | (1.2 S) | | (1.2 S) | | (1.2 S) | |
| Normal + Pipe Rupture (including LOCA) Loads Combination (pressure + deadweight + pipe break/LOCA) | [| | | | | a,c,e] |
| Allowable Stress Limit | (1.8S) | | (1.8S) | | (1.8S) | |
| Thermal Expansion | [| | | | | a,c,e] |
| | $(1.25 \times S_c + 0.25 \times S_h)$ | | $(1.25 \times S_c + 0.25 \times S_h)$ | | $(1.25 \times S_c + 0.25 \times S_h)$ | |

Note:

* DBE loads envelope OBE loads. DBE stresses conservatively evaluated against lower OBE allowable stresses.

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

| Table 5.4-2 | | |
|---|-----------|-------------------|
| RCSES Stress Analysis Summary – SPU Program | | |
| Member | Load Case | Interaction Ratio |
| Steam Generator Frame Structure | 1 | a,c,e |
| | 2 | |
| | 3 | |
| | 4 | |
| RCP Frame Structure | 1 | |
| | 2 | |
| | 3 | |
| | 4 | |
| Steam Generator/RCP Tie Rod | 2, 3 & 4 | |
| Steam Generator Snubber | 2, 3 & 4 | |
| Hold Down Bolts | SG | 1 & 2 |
| | | 3 |
| | | 4 |
| | RCP | 1 & 2 |
| | | 3 & 4 |
| Embedments | SG | 2, 3 & 4 |
| | RCP | 2, 3 & 4 |

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

5.5 Reactor Coolant Pumps and Motors

The reactor coolant pumps (RCPs) at Indian Point Unit 2 (IP2) were evaluated for the stretch power uprate (SPU) in two separate areas: the structural adequacy of the pumps (subsection 5.5.1 of this report), and the acceptability of the RCP motors (subsection 5.5.2).

5.5.1 RCPs Structural Integrity

5.5.1.1 Introduction

This section addresses the ASME Code structural considerations for the pressure boundary components of the Westinghouse Model 93 RCPs. The IP2 RCP equipment specification requires that the design, analysis, materials, welding, inspection, and testing of the pumps meet the requirements of the ASME Code, Section III, 1965 Edition, with Winter 1965, Addenda or later (Reference 1).

The evaluation of the RCPs for the SPU considered the SPU parameters (see Section 2 of this report), and the Nuclear Steam Supply System (NSSS) design transients (see Section 3.1 of this report), which assumed a core power of 3216 MWt.

The evaluation of the RCPs for the SPU compared the operating temperatures and pressures defined in the SPU NSSS parameters to the pressures and temperatures considered in previous analyses of the RCPs. In addition, the NSSS design transients for the SPU were compared to the transients considered in previous evaluations.

5.5.1.2 Input Parameters and Assumptions

The Model 93 RCPs were originally designed and analyzed to meet the equipment specification and the ASME Code. Subsequent evaluations of the RCPs were performed for the 1990 uprate program (Reference 2) and for the 1.4-percent measurement uncertainty recapture (MUR) (Reference 3). The 1990 Urate Program evaluation supplemented the original analyses and was determined to remain bounding and applicable to the 1.4-percent MUR.

The IP2 RCPs are installed in the Reactor Coolant System (RCS) cold leg, between the steam generator outlet and the reactor vessel inlet. The temperatures and pressures used as inputs to the RCP Code structural analysis are those defined for the reactor vessel inlet in the SPU NSSS parameters (Table 2.1-2). The RCPs have been evaluated for the NSSS design transients, as defined for the RCS cold leg by the equipment specification, and updated by the 1990 uprate program, 1.4-percent MUR, and this SPU Program.

The current IP2 SPU Program parameters, NSSS design transients, seismic loadings, nozzle loadings on main and auxiliary nozzles, and auxiliary nozzle transients are either unchanged or remain bounded by the 1990 uprate program.

5.5.1.3 Description of Analysis

Operating Temperature and Pressure

The SPU parameters (see Section 2 of this report) were used to evaluate the acceptability of the RCPs. From the SPU parameters, there are no changes from the current reactor coolant pressure of 2250 psia for any of the SPU cases. The RCS cold-leg temperature (T_{cold}), defined by the vessel inlet (RCP outlet) temperature on the SPU NSSS parameters for the IP2 SPU Program, is a maximum of 538.2°F. The maximum SPU RCS T_{cold} is less than the corresponding 1990 Urate Program T_{cold} temperature of 547.7°F and is also less than the 1.4-percent MUR T_{cold} temperature of 546.7°F. Since none of the temperatures exceeds the previously considered temperatures, and the pressure does not change, the SPU parameters are bounded by those used as inputs to the 1990 Urate Program analyses (Reference 2).

Table 5.5-1 summarizes the cold-leg SPU NSSS temperatures. From Table 5.5-1, the present (1.4-percent MUR) conditions provide a RCS T_{cold} range of 515.5° to 546.7°F, compared to an RCS T_{cold} range of 515.5° to 538.2°F for the SPU Program. Therefore, the present operation range bounds the RCS T_{cold} range for the SPU Program. Furthermore, both conditions are bounded by the 1990 uprate program RCS T_{cold} range of 515.5° to 547.7°F.

Transient Discussion

The NSSS transients for the IP2 SPU are determined in Section 3.1 of this report to have no changes relative to the NSSS design transients defined for the 1.4-percent MUR or the 1990 Urate Program.

The 1990 uprate program NSSS design transients used in the evaluation of the RCP also bound those for the 1.4-percent MUR.

Since there were no changes in the thermal or pressure transients from the 1.4-percent MUR or the 1990 Urate Program, the transients reported in the 1990 Urate Program remain applicable to the IP2 SPU Program.

5.5.1.4 Acceptance Criteria

The acceptance criteria for the ASME Code structural analysis of the RCP pressure boundary are that the analyzed stresses do not exceed the stress allowables of the ASME Code and that the cumulative usage factors from the Code fatigue analysis remain less than 1.0. This can be demonstrated by showing that the design inputs for the SPU Program are either unchanged or bounded by the design inputs used in previous analyses. Note that the previous analyses demonstrate the acceptability of the RCPs.

5.5.1.5 Results

The previous evaluations (1.4-percent MUR and 1990 Uprate Program) have been shown to be applicable to the SPU Program conditions, as discussed in subsection 5.5.1.3 of this report.

The operating temperature and pressure discussion presented in subsection 5.5.1.3 showed that the operating temperatures and pressures are unchanged or bounded by those considered for the 1990 Uprate Program, as shown in Table 5.5-1.

The 1990 Uprate Program temperature and pressure transients and cycle counts are applicable to the SPU Program and shown in Table 5.5-2.

There are no changes in primary stresses (due to no changes in mechanical loadings). Also, the secondary stresses from thermal transients are bounded by the 1990 Uprate Program. The cumulative usage factors are not affected by the SPU Program and the RCPs remain within ASME Code requirements. A summary of the peak stresses and cumulative usage factors is provided in Table 5.5-3.

5.5.1.6 Conclusions

The SPU Program parameters and NSSS design transients have been shown to be bounded by the parameters and transients considered in the 1990 Uprate Program evaluation. Therefore, the conclusions of the 1990 Uprate Program are still valid and applicable to the SPU Program. The RCPs are acceptable from a structural standpoint. The RCP pressure boundary parts still comply with the ASME Code originally specified or later editions. Therefore, the evaluation results of the SPU Program for the RCPs are consistent with and continue to comply with the current licensing basis and acceptance requirements for IP2.

5.5.2 Reactor Coolant Pump Motors

5.5.2.1 Introduction

This section addresses the performance of the RCP motors. The RCP motors are evaluated for the IP2 SPU parameters and best-estimate flows, which assumed a core power of 3216 MWt.

5.5.2.2 Input Parameters and Assumptions

The input parameters considered in the evaluation of the RCP motors are the steam generator outlet temperatures and the flows defined for the IP2 SPU Program.

5.5.2.3 Description of Analysis

The steam generator outlet temperatures and best-estimate flows are considered in a hydraulic analysis using the operating characteristics of the IP2 RCPs. This hydraulic analysis calculates the power requirements for the impeller that operates at the highest cold power. For the IP2 SPU Program, the power requirements from this analysis for hot-loop and cold-loop operation were compared to the power requirements considered for the IP2 1.4-percent MUR evaluation. The power requirements for the SPU Program were determined to be enveloped by requirements for the 1.4-percent MUR. The 1.4-percent MUR was determined to be acceptable based on the RCP motor evaluations for the 1990 Uprate Program in Reference 2. Therefore, the RCP motors are considered acceptable for the SPU based on the 1990 uprate program evaluation.

The 1990 uprate program evaluated the RCP motor loading in three areas:

- Continuous operation at hot-loop temperatures and flows
- Continuous operation at cold-loop temperatures and flows
- Starting across the line with a minimum 85-percent starting voltage

5.5.2.4 Acceptance Criteria

For the IP2 SPU Program, the acceptance of the RCP motor loading is based on the change being enveloped by the loading previously evaluated in the 1990 Uprate Program .

The RCP Equipment Specification requires the motor to drive the pump continuously under hot-loop conditions in accordance with the National Electric Manufacturers' Association (NEMA) Section MG1-20.40-1963 standards.

The motor is also required to start across the line with a minimum 85-percent starting voltage against the reverse flow of the other pumps running at full speed under cold-loop conditions.

5.5.2.5 Results

The 1.4-percent MUR RCP motor evaluation qualified the motors based on the 1990 Uprate Program evaluation (Reference 2) that considered a best-estimate flow of 85,800 gpm. This flow was used with temperatures of 515.5 and 547.4°F at the steam generator outlet, with the case considering the colder temperature producing the limiting conditions. RCP curves developed for the 1990 Uprate Program evaluation show that the pump is operating on the part of the curve for which an increase in flow corresponds to a decrease in power requirements. Thus, an increase in flow from the 85,800 gpm evaluated for the 1990 Uprate Program, while maintaining the same minimum temperature, results in a decrease in motor power required. Since the flow for the SPU has increased and the minimum temperature has been limited to 515.5°F (see Section 2 of this report), the required motor power would decrease and the motor performance is enveloped by the 1990 Uprate Program evaluation in Reference 2.

Therefore, the IP2 RCP motors are acceptable because the parameters for the SPU Program are enveloped by the parameters previously evaluated for the 1990 Uprate Program (Reference 2).

5.5.2.6 Conclusions

The IP2 SPU Program parameters important to the qualification of the RCP motors are the RCS operating temperatures (steam generator outlet), pressures, and the best-estimate flows. Since the previously considered parameters bound the parameters for the SPU Program, including the limit for the minimum steam generator outlet temperature of 515.5°F, the previous 1990 Uprate Program evaluation of RCP motor remains bounding and applicable and the RCP motors are acceptable for operations at the SPU conditions.

5.5.3 References

1. *ASME Boiler and Pressure Vessel Code, Section III, "Rules for Construction of Nuclear Vessels,"* 1965 Edition with Winter 1965 Addenda, The American Society of Mechanical Engineers, New York.
2. *WCAP-11972, Consolidated Edison Company of New York, Inc. Indian Point Unit 2 NSSS Stretch Rating – 3083.4 MWT Licensing Report, Rev. 1, March 1989.*

3. *Indian Point Nuclear Generating Unit No. 2 1.4-Percent Measurement Uncertainty Recapture Power Uprate License Amendment Request Package*, Entergy Nuclear Operations, Inc., November 2002.

| Table 5.5-1 SPU NSSS Conditions Used to Bracket All Operating Conditions | | | | | | |
|---|-----------------------|----------------------|--------------------------------------|----------------------|-----------------------|------------------------|
| | 1990 Uprate Program | | Present (1.4-percent MUR Program) | | SPU Program | |
| Parameter | High T _{avg} | Low T _{avg} | High T _{avg} | Low T _{avg} | High T _{avg} | Low T _{avg} |
| T _{cold} (vessel inlet) | 547.7°F | 515.5°F | 546.7°F | 515.5°F | 538.2°F | 514.0°F ⁽¹⁾ |

| Table 5.5-2 Cold Leg Thermal Transient Summary for RCP Evaluation for IP2 SPU Program ⁽²⁾ | | | |
|--|------------------------------|--------------------------------|-------------|
| | Thermal Transient ΔT (°F) | Pressure Transient ΔP (psi) | Occurrences |
| Normal Condition | | | |
| Heatup/Cooldown | | | a,c,e |
| Unit Loading/Unloading at 5% of Full Power | | | |
| Step Increase/Decrease of 10% Full Power | | | |
| Large Step Load Decrease with Steam Dump | | | |
| Steady-State Fluctuations | | | |
| Upset Condition | | | |
| Loss of Load | | | |
| Partial Loss of Flow | | | |
| Reactor Trip from Full Power | | | |

Notes:

- Limited to a minimum of 515.5°F.
- The transients used in the 1990 Uprate Program are applicable to the current (1.4-percent MUR) and SPU conditions.

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

Table 5.5-3

**RCP Stress and Fatigue Evaluation for
IP2 SPU Program**

| RCP Component | Max Stress Intensity (psi)⁽¹⁾ | Allowable (psi)⁽¹⁾ | Cumulative Usage Factor | ASME Code |
|-----------------------|---|--------------------------------------|--------------------------------|------------------|
| Casing ⁽²⁾ | [] | |] ^{a,c,e} | 1965 |
| Main Flange Bolting | [] | |] | 1965 |

Notes:

1. The ASME code year used as the basis for the allowable stress is listed in the "ASME Code" column of this table.
2. The three values given are for primary general membrane stress intensity, primary membrane plus bending stress intensity, and primary plus secondary membrane plus bending stress intensity. The corresponding allowable stresses are equal to S_m , $1.5 S_m$, and $3 S_m$, where S_m is 16,700 psi.

Bracketed [] ^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

5.6 Steam Generators

Evaluations of the thermal-hydraulic performance, structural integrity, and mechanical hardware have been performed to address operation of the Indian Point Unit 2 (IP2) steam generators at stretch power uprate (SPU) conditions.

5.6.1 Thermal-Hydraulic Evaluation

The thermal-hydraulic evaluations of the IP2 Model 44F steam generator focused on the changes to secondary side operating characteristics at the proposed SPU conditions. The SPU design operating conditions considered are presented in Table 2.1-2 of this report. The evaluations discussed in this section were performed to confirm the acceptability of the steam generator secondary side parameters. Four cases were analyzed at the 3230-MWt Nuclear Steam Supply System (NSSS) power corresponding to the 3216-MWt core power conditions: 2 Reactor Coolant System (RCS) primary average temperatures (T_{avg}), 548.9 and 571.9°F; and 2 steam generator tube plugging (SGTP) levels, 0 and 10 percent. Each case was evaluated for 2 feedwater temperatures: 436.2 and 390°F. The high feedwater temperature cases are referred to as Case "a" (such as Case 2a), while the low temperature cases are referred to as Case "b" (such as Case 2b) for this section of the report. The steam generator secondary side operating characteristics at the SPU conditions are compared with a reference case for the 3083.4-MWt NSSS power condition resulting from the 1990 Uprate Program. The results of the thermal-hydraulic evaluations are summarized in Table 5.6-1. Based on these evaluations, the IP2 steam generators are qualified to operate at the SPU conditions with up to 10-percent SGTP.

Methodology

A number of secondary side operating characteristics were considered to assess the acceptability of steam generator operation at various operating conditions. These operating characteristics include bundle mixture flow rate, steam pressure, average heat flux and margin to departure from nucleate boiling (DNB) transition, moisture carryover (MCO), hydro-dynamic instability, secondary inventory, and secondary side pressure drop. Numerical values of parameters representing the above operating characteristics obtained using the GENF code are shown in the lower half of Table 5.6-1.

GENF is a one-dimensional (1-D), steady-state thermal and hydraulic performance code developed by Westinghouse specifically for feed-ring steam generators. The code has been verified and is maintained under Westinghouse Configuration Control. GENF calculates the overall primary side heat balance based on the thermal power, primary flow rate, and the primary outlet temperature and operating pressure. On the secondary side, the code

determines the secondary side saturation pressure in the tube bundle using an iterative procedure. The steam outlet pressure is then calculated by subtracting all losses from the bundle region to the steam nozzle outlet. The steam outlet pressure is used to determine steam flow rate via the secondary side heat balance and feedwater inlet temperature.

An iterative calculation is performed to determine the circulation ratio and various secondary side pressure drops. Finally, the fluid masses and volumes, as well as the stability damping factor are calculated. The stability damping factor is a measure of stable operation of the steam generator.

The ATHOS code was used to evaluate the potential for local tube dryout or margin to DNB. ATHOS is a three-dimensional (3-D) computer program for analyzing the computational fluid dynamics of steam generators. The ATHOS code was developed under the sponsorship of the Electric Power Research Institute (EPRI). The ATHOS code consists of a geometry pre-processor, a thermal-hydraulic (ATHOS) solver, and a post-processor module. The geometry pre-processor simulates the detailed geometry. This geometry simulation includes the detailed tube layout, tube lane blocks, flow distribution baffle, tube support plates (TSPs), anti-vibration bars (AVBs), and opening of the primary separators. The ATHOS module uses the pre-processor data to calculate the primary and secondary side thermal-hydraulic parameters in the steam generator. The ATHOS code calculates both heat flux and tube wall temperature in addition to typical parameters such as liquid velocity, vapor velocity, and steam quality for a two-phase flow like that in the secondary side of a steam generator.

The ATHOS code for the analysis of steam generators has been verified and qualified by EPRI and Westinghouse. The Westinghouse-developed post-processors process the large amount of output from the ATHOS calculation. Their capabilities include: velocity vector plots; and contour plots of thermal-hydraulic parameters, such as steam quality, velocity, heat flux, and critical steam quality corresponding to DNB.

Bundle Mixture Flow Rate

The steam flow rate increases proportionally with power when operating with the same T_{avg} and feedwater temperature. With the SPU power level, the calculated steam flow rate per generator increases from []^{a,c,e} million lb/hr and the circulation ratio decreases from []^{a,c,e}. Since the tube bundle mixture flow rate is the product of the circulation ratio and the steam flow rate, the resulting bundle flow rate is approximately []^{a,c,e} lb/hr for all cases, or essentially the same at both 3083.4 MWt and 3230 MWt.

The secondary fluid velocities in the U-bend region at the SPU conditions are approximately []^{a,c,e} percent higher without tube plugging, and []^{a,c,e} percent higher with 10-percent

SGTP, depending on the feedwater temperature. The fluid velocities in the downcomer and at the wrapper opening are up to []^{a,c,e} percent lower at the SPU 548.9°F steam generator average temperature condition with 10-percent SGTP, and approximately []^{a,c,e} percent higher for the 571.9°F steam generator average temperature condition with 0-percent SGTP. The SPU and the small changes in T_{hot} and feedwater temperatures essentially have no significant effect on the secondary flow both in the downcomer and in the tube bundle.

Steam Pressure

Steam pressure is affected by both the available heat transfer area in the tube bundle and the average primary fluid temperature. With the SPU and steam generator average temperature of 548.9°F, and 0-percent SGTP, the steam pressure calculated using the Westinghouse GENF code shows a decrease from []^{a,c,e} psia. Increasing the SGTP to 10 percent results in a further reduction of steam pressure to []^{a,c,e} psia. Note that these are theoretical values and the plant operations will limit minimum normal operating full-power steam pressure to 650 psia.

Heat Flux

Average heat flux in the steam generator is directly proportional to heat load and inversely proportional to the heat transfer area in service. For the 0-percent SGTP, the average heat flux increases from []^{a,c,e}. With 10-percent SGTP and SPU conditions, the average heat flux increases to []^{a,c,e} BTU/hr-ft².

A measure of the margin for DNB transition in the bundle is a check of the ratio of the local quality, to the estimated quality at DNB transition, or (X/X_{DNB}) . The ATHOS code was used to estimate the (X/X_{DNB}) ratio for the SPU conditions. ATHOS analyses show that the maximum (X/X_{DNB}) at 3230 MWt with 0-percent SGTP is []^{a,c,e}, which is significantly below a ratio of 1.0 at which point dryout is expected. With the largest tube plugging level of 10 percent and 3230 MWt, the maximum (X/X_{DNB}) ratio predicted is []^{a,c,e}. This case was analyzed with a conservative assumption that all the plugged tubes are concentrated around the bundle outer periphery where heat flux is relatively low. Normally, in an operating unit, the plugged tubes would be randomly distributed, and the maximum (X/X_{DNB}) ratio would be expected to be less than that obtained assuming that all plugged tubes are located at the outer periphery. Therefore, tube wall dryout is not expected to occur at the SPU conditions.

Moisture Carryover

The performance of moisture separator packages is primarily determined by three operating parameters: steam flow (power), steam pressure, and water level. For the moisture separator performance data evaluation, steam flow and steam pressure are combined into a single parameter designated as the separator parameter (SP). A correlation for MCO as a function of SP was used to predict MCO at desired conditions. The values of SP for the IP2 power SPU conditions were calculated using the GENF code results.

Earlier MCO projections for IP2 replacement steam generators (RSGs) were performed using data from several other plants with the same modular separator package. A field test was performed in 2001 to measure MCO for IP2 steam generators. The average MCO for all steam generators from this test lies above the regression obtained by adding the IP2 data to the existing database, as shown in Figure 5.6-1. The magnitude of the deviation is comparable to the scatter found in the other data points. However, as a conservative measure, a line parallel to the regression line and passing through the 2001 IP2 test data was used to project MCO for the SPU conditions.

The calculated MCO increases from []^{a,c,e} percent of steam flow at 3083.4 MWt to a maximum of []^{a,c,e} percent at 10-percent SGTP and SPU conditions. Even this conservative evaluation predicts that the MCO at the SPU condition will be below the 0.25-percent limit.

Hydro-Dynamic Stability

The hydro-dynamic stability of a steam generator is characterized by its damping factor. A negative value of the damping factor indicates that any disturbance to thermal-hydraulic parameters (for example, flow rate or water level) will rapidly reduce in amplitude, and the steam generator will return to stable operation. The SPU by itself has no significant effect on the damping factor, however, at the reduced feedwater temperature of 390°F, the damping factor decreases from []^{a,c,e}. Even the lowest damping factor calculated []^{a,c,e} is substantially negative. Therefore, the IP2 steam generators will continue to operate in a hydro-dynamically stable manner at the SPU operating conditions.

Steam Generator Secondary Fluid Inventory

Secondary side fluid inventory consists of the masses of the liquid and the vapor phases. The vapor mass is 5 to 6 percent of the total inventory. With the proposed SPU, at 0-percent SGTP, the secondary fluid mass decreases from []^{a,c,e} lbs—a change of about 2 percent. The minimum calculated inventory of []^{a,c,e} lbs would occur for the SPU case

with 10 percent of the tubes plugged. The small changes in inventory are judged to have no effect on operation.

Steam Generator Secondary Side Pressure Drop

The secondary side pressure drop (from the feedwater nozzle to the steam exit nozzle) increases from []^{a,c,e} psi as a result of the SPU. The largest pressure drop, []^{a,c,e} psi, is predicted with 10-percent SGTP, for T_{avg} of 548.9°F and SPU operating conditions. The small increase in pressure drop should have no significant effect on the feed system operation.

Thermal-Hydraulic Evaluation Conclusion

In conclusion, the thermal-hydraulic characteristics of the IP2 Model 44F steam generators are within acceptable ranges for the SPU conditions with a tube plugging level below 10 percent.

5.6.2 Structural Integrity Evaluation

The structural evaluation for the SPU focused on the critical steam generator components. The critical components are those that are currently the most highly stressed, or have the highest cumulative fatigue usage, for the present operating conditions. The critical components are affected by changes in the pressure and temperature in the primary and secondary side of the steam generator. The following critical primary side components were evaluated: divider plate, tubesheet and shell junction, tube-to-tubesheet weld, and tubes. The critical secondary side components included: feedwater nozzle, secondary manway bolts/studs, and steam nozzle.

Comparisons of the primary side transients and RCS parameters were performed to determine the scale factors that were applied to the baseline analyses maximum stress ranges and fatigue usage factors. The scale factor was applied to the baseline analysis results for various components to obtain the values for the SPU conditions.

For the primary side components (particularly the divider plate, the tubesheet and shell junctions, the tube-to-tubesheet weld, and tubes), the applicable scale factors were the ratios of the primary-to-secondary side differential pressure for the baseline and SPU conditions.

The scale factors are applied to the stress ranges that are a combination of both thermal and pressure effects, and the revised stress ranges are used to calculate the revised alternating stress and the fatigue usage.

For the secondary side components, such as the feedwater nozzle and secondary manway bolts/studs, the decrease in secondary side pressure was the basis for the scale factors. The

reduced pressure results in an increased stress range during transient events. The increase in stress range due to the reduced pressure is added to the baseline stress range to evaluate the revised stress range and the revised fatigue usage.

Input Parameters and Assumptions

The SPU structural evaluation was performed for 3230-MWt NSSS power and 10-percent SGTP. The applicable design operating conditions used for the steam generator's structural evaluation are shown in Table 2.1-2. The design transients and the results of the steam generator primary-to-secondary side ΔP calculation (discussed in subsection 5.6.3), were used to generate scaling factors with respect to the design basis stress reports results. The scaling factors were based on the steam temperature of 513.5°F, corresponding to a steam pressure of 768 psia.

The plant operating conditions provide for both a low T_{avg} temperature operating condition case and a high T_{avg} temperature operating condition (see Table 2.1-2). The low T_{avg} case results in a lower steam pressure (and, therefore, a greater primary-to-secondary side ΔP) and, as such, will envelop the high T_{avg} case. On this basis, the bounding scale factors based on low T_{avg} were used to perform a bounding evaluation of the critical components.

The scale factors calculated based on the differential pressure are applied conservatively on the stress intensities that are due to both pressure and thermal loads since the scale factors based on pressure will envelop the scale factors based on thermal loads. The evaluation based on scale factors due to ΔP is conservative since the thermal variation is small.

Description of Analyses and Evaluations

This structural evaluation was performed for the bounding condition for low T_{avg} case, where $P_{stm} = 650$ psia. The existing design basis evaluation corresponds to the reference of $P_{stm} = 768$ psia. Scale factors are calculated based on the revised steam pressure at the SPU operating conditions.

Primary Side Components

For primary side components, the scale factor is based on the change in the primary-to-secondary side differential pressure and was calculated based on the following equation:

$$\left[\frac{\Delta P_{primary-to-secondary}}{\Delta P_{design}} \right]^{a,c,e}$$

Secondary Side Components

Secondary side components, such as the feedwater nozzle and secondary manway studs, are subjected to only the steam pressure. Therefore, the scale factor is calculated based on the reduced steam pressure during transient events.

The calculated scale factors were applied to the stress ranges for all applicable transient combinations involved in the original reference analysis.

Applying the scale factors to the design basis stresses approximates the stress and fatigue usage values that would occur during operation at the SPU conditions.

Acceptance Criteria

The acceptance criteria for each component are consistent with the criteria used in the design basis analysis for that component. The maximum range of primary-plus-secondary stresses was compared with the corresponding $3S_m$ limits (Reference 1). For situations in which these limits were exceeded, a plastic analysis or simplified elastic-plastic analysis was performed consistent with the original design basis analysis.

A cumulative fatigue usage factor less than or equal to unity demonstrates the adequacy of the steam generators for a 40-year design life.

Results

The results of the evaluation show that all components analyzed meet American Society of Mechanical Engineers (ASME) Code Section III limits for a 40-year design life.

The fatigue usage of the secondary manway studs without SPU is equal to []^{a,c,e}. For operation at the SPU conditions, the fatigue usage would be []^{a,c,e}. The fatigue usage after uprating remains less than the maximum value of 1.0, indicating that the secondary manway studs are adequate for 40 years of service.

The results of the evaluation are summarized in Table 5.6-2.

5.6.3 Evaluation of Primary-to-Secondary Side Pressure Differential

An analysis was performed to determine if ASME Boiler and Pressure Vessel (B&PV) Code, 1965 Edition through Summer 1966 Addenda (Reference 1) limits on the Model 44F RSG design primary-to-secondary ΔP are exceeded for any of the applicable transient conditions, for

the SPU parameters (Table 2.1-2). The design pressure limit for primary-to-secondary pressure differential is 1700 psi, as defined in the applicable design specification.

The normal/upset transient conditions are subject to the following design pressure requirements:

- **Normal Condition Transients:** Primary-to-secondary pressure gradient should be less than the design limit of 1700 psi.
- **Upset Condition Transients:** If the pressure during an upset transient exceeds the design pressure limit, the stress limits corresponding to design conditions apply using an allowable stress intensity value of 110 percent of those defined for design conditions. In other words, as long as the upset condition pressure values are less than 110 percent of the design pressure values, no additional analysis is necessary. For the IP2 steam generators, 110 percent of the design pressure limit corresponds to 1870 psi.

The primary-to-secondary pressure differential evaluation was based on the transient parameters discussed in Section 3.1 of this report and the corresponding full-power conditions that are defined in Table 2.1-2 of this document. The pressure differentials across the primary-to-secondary side pressure boundary are calculated for these defined full-power conditions. Note that the evaluation was performed for the 10-percent SGTP condition since increased levels of plugging result in greater primary-to-secondary pressure differentials.

The analysis determined that the maximum normal/upset operating condition primary-to-secondary side differential pressures for high T_{avg} operation would be []^{a,c,e} psi for normal operating condition transients, and []^{a,c,e} psi for upset condition transients. For the low T_{avg} operating conditions, the maximum pressure differentials are []^{a,c,e} psi for the normal and upset conditions, respectively. The results show that the maximum primary-to-secondary pressure gradients are less than the allowable values of 1700 and 1870 psi for normal and upset operating conditions, respectively. Therefore, the design pressure requirements of the ASME Code continue to be satisfied.

5.6.4 Evaluations for Repair Hardware

The IP2 RSGs entered service in 2000. During the fabrication on 1 of the steam generators, several Westinghouse shop welded plugs were installed. These components were re-evaluated for the operating conditions and transients associated with SPU operation.

In anticipation of future needs, both "long" and "short" 7/8-inch ribbed mechanical plugs were qualified for installation in the Model 44F RSGs for the SPU operating conditions. In addition, since there are circumstances that may require tube ends to be reamed, a 40-percent tube wall undercut was considered. The resulting reduced tube mouth weld joint geometry is qualified for continued service. Also, in case a steam generator tube requires stabilization in the future, evaluations were performed to qualify collar-cable tube stabilizers and bare-cable stabilizers.

Mechanical Plugs

The enveloping condition for the Westinghouse mechanical plug (Alloy 690 plug shell material) results in the largest pressure differential between the primary and secondary sides of the steam generator. Both the SPU parameter changes and the updated NSSS design transients were used to determine the effect of the power uprating on the mechanical plugs. The most critical set of parameters for the mechanical plug evaluation are those for the primary side hydro-static pressure test in which the differential pressure across the plug is []^{a,c,e} psi and is independent of power uprate.

Description of Evaluation

A structural evaluation was performed for both "long" and "short" Westinghouse 7/8-inch ribbed mechanical plugs for both the 1.4-percent MUR and the SPU conditions. This evaluation was performed to the applicable requirements of ASME B&PV Code (Reference 1).

Acceptance Criteria

The Westinghouse mechanical tube plug was evaluated for the original NSSS design transients and for the changes to these transients due to the SPU. The primary stresses due to design, normal, abnormal, and test conditions must remain within the respective code allowable values (Reference 1). The maximum range of primary-to-secondary stresses is limited to $3S_m$. The cumulative fatigue usage factor must be less than or equal to 1.0, or the ASME fatigue exemption rules must apply for a 40-year fatigue life for the plug. In addition to the stress criteria, plug retention must be ensured.

Results

The critical loading parameter from the design of the plug shell is the primary pressure. The plug qualification was based on a primary pressure of 2485 psig. The maximum design primary-to-secondary differential pressure of 1700 psi for plug retention was also addressed.

All stress/allowable ratios are less than unity, indicating that all primary stress limits are satisfied for the plug shell wall between the top land and the plug end cap. The plug meets the Class 1 fatigue exemption requirements per N-415.1 of the ASME Code (Reference 1).

Since this is a component that is installed into the steam generator after original fabrication is complete, and since this part is typically fabricated to the requirements of the 1989 ASME Code Edition (Reference 2), an evaluation was conducted based on the 1989 code year requirements. It was determined that the mechanical plug is also acceptable for the SPU operating conditions based on the 1989 ASME Code Edition.

Conclusions

Results of the analyses performed for the mechanical plug for the IP2 show that both the long and short mechanical plug designs satisfy all applicable stress and retention acceptance criteria at the SPU condition with up to 10-percent tube plugging. Note that the mechanical plugs have been previously qualified for the SPU condition with up to 25-percent tube plugging.

Shop Weld Plugs

The Westinghouse shop-welded plugs are fabricated from ASME SB-166, Alloy 600 rod material. The minimum yield for this material is 35,000 psi. Since several design transients were revised for the SPU conditions, a revised analysis was performed to qualify the plugs for the revised conditions.

Description of Evaluation

A structural evaluation was performed for the existing shop-welded tube plugs for the SPU operating conditions. The applicable design transients were also addressed. The evaluation was performed to the applicable requirements of the ASME B&PV Code (Reference 1).

Acceptance Criteria

The primary stresses due to design, normal, abnormal, and test conditions must remain within the respective ASME Code allowable values (Reference 1). The maximum primary-to-secondary stresses are limited to $3S_m$. The cumulative fatigue usage factor must be less than or equal to 1.0, or the ASME fatigue exemption rules must apply for a 40-year fatigue life for the plug.

Results

The evaluation of the weld plug first addressed the design condition. A vertical minimum weld thickness critical plane around the perimeter (circumference) of the weld plug was considered. The design pressure differential of 1700 psi between the primary and secondary was applied to the plug.

Test conditions for the primary hydro-static and secondary hydro-static tests were then evaluated. Values for primary stresses, primary stresses plus secondary stresses, and primary-to-secondary stress range intensities were calculated. All stress values were determined to be acceptable.

The normal and abnormal conditions were then reviewed. It was determined that the controlling transient for both the normal and abnormal conditions was the "steady-state fluctuation" transient. The differential pressure considered was []^{a,c,e} psi. This was the controlling pressure condition for the SPU transient conditions. It was determined that the stress limits are acceptable for the controlling differential pressure.

The last step in the evaluation process considered fatigue. The approach was to investigate if the weld plug would be exempt from an explicit usage factor calculation based on the ASME requirements for fatigue exemption. The 6 required fatigue exemption conditions were determined to be satisfied. Therefore, it was concluded that the welded plug does meet the ASME Code cycle load fatigue limits for the SPU.

Conclusions

All primary stresses are satisfied for the weld between the weld plug and the tubesheet cladding. The overall maximum primary-plus-secondary stresses for the enveloping transient case of "steady-state fluctuation" were determined to be acceptable. The fatigue evaluation for the weld plug used the ASME fatigue exemption rules. It was determined that the fatigue exemption rules were met and, therefore, fatigue conditions are acceptable.

Tube Undercut Qualification

Steam generator tube-mouth-end field machining may be required to implement modifications and tube repair (that is, plugging, sleeving, and tube end reopening). It is sometimes necessary to remove a portion of the tube and weld material with a machining process (drilling and reaming) when removing a Westinghouse mechanical plug. The structural evaluation performed for the SPU conditions addressed the acceptability of up to a 0.020-inch (40 percent of the 0.050-inch tube wall) undercut of the tube wall thickness. The evaluation was performed to the applicable requirements of ASME B&PV Code (Reference 1).

Description of Evaluation

Past structural evaluations for steam generator tube end machining have been performed for various steam generator models. The approach for the IP2 tube end evaluation was to use the results from a previous evaluation and adjust the stress values from the past project as appropriate for the original NSSS design transients and for changes to these transients due to uprating. The adjustment value was conservatively based on the tubesheet geometry and tube hole pitch.

Acceptance Criteria

The primary stresses due to design must remain within the respective ASME Code allowable values (Reference 1). The maximum range of stress intensities is limited to $3S_m$. The cumulative fatigue usage factor must be less than or equal to 1.0, or the ASME fatigue exemption rules must apply for a 40-year fatigue life for the tube undercut. A similar approach, using stress factors, was used in the investigation of fatigue for the tube undercut machining.

Results

The results determined that all revised stresses for the SPU condition are within ASME Code allowable values.

It was determined that fatigue usage values, when adjusted for the SPU condition, remain acceptable.

Conclusions

The results of the stress evaluation of the IP2 model 44F steam generators determined that the stresses are within ASME Code allowable values. Also, the fatigue usage factors were determined to remain less than 1.0.

Collar-Cable-Stabilizer Qualification

The Westinghouse collar-cable stabilizer consists of a central coaxial cable made up of Type 302 stainless steel wire strands protected over the full length of the stabilizer by several Type 304 stainless steel tubular collars, which are swaged onto the cable. The swaged collars are about 8 inches long with a longitudinal space of about 1 inch between the adjacent collar segments. This arrangement provides flexibility and dynamic damping.

Description of Evaluation

The qualification method was employed to show that the wall of an assumed hypothetical fully severed host tube would wear out before the stabilizer collar wore away should a random wear couple form between the severed host tube and the stabilizer collar. Under these conditions, the central coaxial cable of the stabilizer would remain intact and protected by the collar remnant for the life of the installation. The evaluation approach was based on the relative wear coefficients and cross-sectional areas of the tube and stabilizer, and was independent of the dynamic fluid forces causing potential random vibration of the assumed severed host tube.

Acceptance Criteria

The design intent of the Westinghouse cable stabilizer is that the local tube wall wears out totally before the tubular segment of the stabilizer wears out, thereby providing positive protection from wear of the stabilizer's central co-axial cable for the life of the installation. Also, the worn stabilizer remnant should prevent significant contact with the adjacent tubes.

Results

The qualification was based solely on geometric parameters and the relative wear coefficients between the stabilizer collars and the host tube materials. Should a potentially unstable dynamic condition occur and the tube starts to wear against the stabilizer collar, the tube wall was determined to essentially wear through before the collar wears through (which protects the central co-axial cable for the life of the installation). Also, potentially deleterious contact with adjacent active tubes was determined not to occur.

Conclusions

The evaluation of the straight leg collar-cable tube stabilizer for IP2 model 44F steam generators determined that the 0.625-inch diameter stabilizer is acceptable for use in the 0.875-inch diameter, 0.050-inch nominal wall tubes, for operation at the SPU conditions.

Bare-Cable Stabilizer Qualification

The Westinghouse bare-cable stabilizer's function is to retain severed tubes, to dampen vibration and to mitigate additional wear on plugged steam generator tubes. The tube stabilizer is fabricated from 0.5-inch diameter 6 x 19 Type 304 stainless steel wire rope. It has a lower end fitting that allows it to be installed with a typical probe pusher. The upper end of the stabilizer is capped with a welded bullet nose to facilitate installation. This stabilizer is used in the same manner as the collar cable stabilizer discussed previously.

Description of Evaluation

It has been previously demonstrated that the bare-cable tube stabilizer is acceptable generically for use in Westinghouse-designed steam generators with 7/8-inch tubing. The generic design was based on the original Sequoyah Units 1 and 2 steam generators and is generally applied to defects below the first tube support. However, longer lengths of this stabilizer design can be applied to defects anywhere along the straight length of the tubing. A review of the generic bare-cable stabilizer analysis and the SPU thermal-hydraulic conditions shows that the existing qualification of the stabilizer remains valid for the SPU conditions at IP2.

Both IP2 and Sequoyah have similar Westinghouse steam generator designs. The tube support geometry for both designs is essentially the same except the Model 44F steam generators at IP2 have a flow distribution plate located approximately 23 inches up from the secondary face of the tubesheet. However, this plate is not assumed to provide any support for the tubes. Thus, the free-span region of the tube at the entrance to the tube bundle is essentially the same for both steam generator designs. Other assumptions used in the generic bare-cable stabilizer analysis (for example, threshold instability constant, tube inside diameter [ID] and outside diameter [OD], damping ratio, etc.) are the same for both Sequoyah and IP2.

Comparisons of the SPU operating conditions for IP2 and those considered in the qualification of the stabilizers for Sequoyah were used to determine the applicability of the generic analysis to IP2.

Acceptance Criteria

The bare-cable stabilizer design is considered qualified if the tube with the stabilizer installed remains fluid-elastically stable for operation at the SPU conditions. That is, the stability ratio of a tube with a stake must be less than or equal to 1.0.

Results

A review of the thermal-hydraulic analysis shows that the SPU results in a maximum increase in fluid velocities at the tube bundle entrance of no more than 3 percent. More significantly, the dynamic pressure (ρV^2) of the fluid against the tubes increases by approximately 4 percent for the worst case analyzed. To account for these potential differences, the previous generic bare-cable stabilizer evaluation included secondary side flow velocities increased by 50 percent and the unsupported tube span at the tube bundle entrance lengthened by 25 percent. Even with these overly conservative assumptions, the stability ratio remains less than 1.0, and the tube movements would be less than the defined limits. Thus, the existing qualification for the bare-cable stabilizer is bounding for the SPU operating conditions proposed for the IP2 steam generators.

Conclusions

The bare-cable tube stabilizer is acceptable for use in the IP2 steam generator at the SPU conditions.

Structural Evaluation Conclusions

Results of the analyses performed on the IP2 Model 44F RSGs show that all steam generator components continue to meet ASME B&PV Code Section III, "Rules for Construction of Nuclear Vessels," 1965 Edition, through Summer 1966 Addenda (Reference 1) limits for the SPU conditions. The primary-to-secondary pressure differential remains below the design value of 1700 psi for normal operating conditions, and 1870 psi for upset conditions. In addition, weld plugs, mechanical plugs, cable-collar stabilizers, and bare-cable stabilizers are qualified for use in the Model 44F RSGs at the SPU conditions.

5.6.5 Regulatory Guide 1.121 Analysis

The heat transfer area of steam generators in a pressurized water reactor (PWR) NSSS comprises over 50 percent of the total primary system pressure boundary. The steam generator tubing, therefore, represents a primary barrier against the release of radioactivity to the environment. For this reason, conservative design criteria have been established for the maintenance of tube structural integrity under the postulated design basis accident (DBA) condition loadings in accordance with Section III of the ASME Code.

Over a period of time under the influence of the operating loads and environment in the steam generator, some tubes may become degraded in local areas. Partially degraded tubes are satisfactory for continued service as long as the defined stress and leakage limits are satisfied, and as long as the prescribed structural limit is adjusted to account for possible uncertainties in the eddy current inspection and an operational allowance for continued tube degradation until the next scheduled inspection.

The Nuclear Regulatory Commission (NRC) Regulatory Guide (RG) 1.121 (Reference 3) describes an acceptable method for establishing the limiting safe condition of tube degradation beyond which tubes determined to be defective by the established in-service inspection should be removed from service. The level of acceptable degradation is referred to as the "repair limit." For tube cracking due to fatigue and/or stress corrosion, a specification on maximum allowable leak rate during normal operation must be established so that a reasonable likelihood that "leak before break" would be achieved. If the leak rate exceeds the specification, the plant must be shut down and corrective actions taken to restore the integrity of the unit. The EPRI *PWR*

Primary-to-Secondary Leak Guidelines (Reference 4) form the basis of the plant's operational leakage program.

Description of Evaluation

An analysis has been performed to define the "structural limits" for an assumed uniform thinning mode of degradation in both the axial and circumferential directions. The assumption of uniform thinning is generally regarded to result in a conservative structural limit for all flaw types occurring in the field. The allowable tube repair limit, in accordance with RG 1.121 (Reference 3), is obtained by incorporating a growth allowance for continued operation until the next scheduled inspection and also an allowance for eddy current measurement uncertainty into the resulting structural limit. Calculations have been performed to establish the structural limit for the tube straight leg (free-span) region of the tube for degradation over an unlimited axial extent, and for degradation over limited axial extent at the tube support plate (TSP), flow distribution baffle (FDB), and anti-vibration bar (AVB) intersections.

Results and Conclusions

A summary of the tube structural limits as determined by this analysis for both the high T_{avg} and low T_{avg} operating conditions is provided in Table 5.6-3. The corresponding repair limits are established by subtracting from the structural limits an allowance for eddy current uncertainty and continued growth. The reduced minimum tube wall thickness (t_{min}) requirements established for the AVB intersections in Table 5.6-3 only apply for tube rows 14 and higher. The t_{min} requirements and structural limits corresponding to the FDB are to be used for AVB intersections in tube rows 1 to 13.

5.6.6 Tube Vibration and Wear

The effect of the proposed SPU on the steam generator tubes was evaluated based on the current design basis analysis and included the changes in the thermal-hydraulic characteristics of the secondary side of the steam generator resulting from the SPU. The effects of these changes on the fluid-elastic instability ratio and amplitudes of tube vibration due to turbulence have been addressed. In addition, the effects of the SPU on potential future tube wear have also been considered.

Description of Analyses and Evaluation

The baseline tube vibration and wear analysis results for the IP2 Model 44F RSG were used for comparison. The original vibration analysis demonstrated that the maximum fluid-elastic stability ratio for the expected tube support conditions was less than the allowable limit of 1.0. The original tube vibration analysis also determined that negligible tube responses occurred due

to the vortex-shedding mechanism. The amplitudes of vibration due to turbulence were also determined to be reasonably small with maximum displacements that were determined to be on the order of a few mils []^{a,c,e}. The maximum expected tube wear that could occur over the remaining period of operation was calculated to be []^{a,c,e}.

The results of the vibration and wear analysis were modified to account for anticipated changes in secondary side operating conditions due to the SPU. The following is a summary of results.

For the expected support conditions, it was determined that straight leg stability ratios were not significantly affected. However, the stability ratio for U-bend conditions increased from []^{a,c,e}, which is still less than the allowable limit of 1.0. As a result, the analysis indicated that large amplitudes of vibration are not projected to occur due to the fluid-elastic mechanism while operating the steam generator in the SPU condition.

The maximum displacement values calculated for turbulence excitation in the original analysis were modified to account for SPU-induced changes in the operating conditions. For the most limiting tube support condition, it was determined that the turbulence-induced displacement could increase from []^{a,c,e}. Displacements of this magnitude are not sufficient to produce tube-to-tube contact. However, the potential for tube wear must be considered.

As in the original analysis, the vortex-shedding mechanism was determined not to be a significant contributor to tube vibration, which continues to be the case for operation in the post-SPU condition.

The potential for tube wear was addressed in the original analysis that addressed wear in both the straight leg and U-bend portions of the steam generator. These calculations were then updated to reflect operation of the steam generator in an uprated condition. The SPU calculation determined that the level of tube wear that could occur would increase from []^{a,c,e} at the SPU conditions. From these calculations it can be concluded that although there may be an increase in the level of wear that could occur at the SPU operating conditions, the increased level would not be significant. Any increase in the rate of tube wear would progress over many cycles and would be observable during normal eddy current inspections, at which time remedial action could be taken.

Thirteen tubes were identified with AVB wear during the steam generator inspections for refueling outage 15 (RFO-15). A 100-percent inspection of the 4 steam generators was performed. All 13 tubes were administratively plugged during that outage even though no tube had reported wear greater than 20-percent through-wall. The small number of tubes with AVB wear were judged to be outliers and not typical of the general tube behavior. The existing

baseline analysis was therefore judged to be representative of all of the active tubes that show no wear.

However, in light of the recent inspection results that indicate that some level of tube wear, albeit small, is occurring in the pre-uprate condition, it would not be unreasonable to expect that some level of tube wear could also potentially occur in the post-uprate operating condition. Should any additional tube wear actually occur after the uprate, the rate of tube wear would be reasonably small and would not be expected to result in any tubes wearing down to a minimum allowable tube wall thickness in 2 cycles of operation. Any wear that would occur would be expected to be limited to a small population of tubes in the AVB region. As a result, it would be prudent to eddy-current inspect the tubes in the U-bend region of the steam generator within the next two cycles after the uprate in power is implemented. This inspection would be sufficient to determine the degree of wear, should it occur, and allow sufficient time to perform any necessary remedial actions.

There is no direct correlation of flow-induced vibration with primary-to-secondary side pressure differences. The steam generator tubes respond primarily to the conditions associated with the secondary side since the forcing functions associated with the secondary side of the steam generator dominate any other effects. Any effects of primary-to-secondary side pressure difference are inherently considered in the analysis in that the secondary side conditions are defined by the total steam generator conditions such as steam pressure, flow rates, re-circulation, etc., and include the primary-to-secondary side pressure difference.

In some model steam generators, particular consideration is given to the potential for high cycle fatigue of U-bend tubes. This phenomenon has been observed in tubes with carbon steel support plates where denting or a fixed tube support condition has been observed in the upper most plate. However, since the IP2 steam generator TSPs are manufactured from stainless steel, there is no potential for the necessary boundary conditions (that is, denting) to occur at the uppermost support plate. Hence, high cycle fatigue of U-bend tubes is not an issue for the IP2 Model 44F steam generators.

Conclusions

The analysis of the IP2 Model 44F RSGs indicates that significant levels of tube vibration will not occur from the fluid-elastic, vortex shedding, or turbulent mechanisms as a result of the proposed SPU. In addition, the projected level of tube wear as a result of vibration can be expected to remain small and not result in unacceptable wear.

5.6.7 Tube Integrity

Over a period of time, some tubes can become degraded locally under the influence of the operating loads and chemical environment in the steam generator. Degradation mechanisms observed in the first generation steam generators (for example, those using mill annealed [MA] Alloy 600 tubing) include OD stress corrosion cracking (ODSCC), primary water stress corrosion cracking (PWSCC), pitting, as well as tube wear at AVBs and TSPs due to tube vibration, and potentially at other locations such as the FDB, due to maintenance operations. The potential for these degradation mechanisms affecting the IP2 steam generators due to the SPU is discussed below.

The IP2 steam generators are Model 44F steam generators that use Alloy 600TT (thermally treated) tubes, and other design features (discussed below) that minimize the potential for tube degradation. Comparative studies (for example, EPRI TR-108501 [Reference 5]) of the performance of Alloy 600TT and Alloy 600MA have shown Alloy 600TT has superior resistance to corrosion compared to Alloy 600MA. Plants using Alloy 600TT have operated without evidence of PWSCC for over 15 effective full-power years (EFPYs) at hot leg operating temperatures of up to 618°F. The IP2 hot leg operating temperature is expected to be limited to 605.8°F (reduced from the currently approved temperature of 611.7°F) following an NSSS power uprating to 3230 MWt.

Secondary side steam generator chemistry has contributed to tube cracking in some units. Concentration of caustic solutions in areas of stress concentration aids the initiation of cracking. Stress corrosion cracking of Alloy 600 tubing is believed to follow an Arrhenius relationship, therefore, the reduction of maximum temperatures in the steam generator (T_{hot}) should decrease the propensity for development of stress corrosion cracking.

ODSCC was reported in a plant with Alloy 600TT tubing in May 2002 after about 9.7 EFPYs of operation. The cause for the ODSCC in this plant has not yet been confirmed, but is believed to be attributed to an off-nominal tube material condition. The presence of the condition is believed to be observable using bobbin-coil eddy current inspection. Thus, if any tubes in the IP2 steam generators contain a similar material condition, these tubes can be identified and effectively monitored by nondestructive examination (NDE).

In addition to enhanced tube materials of construction, the IP2 steam generators use design features that have been shown to effectively reduce the potential for stress corrosion cracking (SCC) initiation. These include; hydraulically expanded tubes in the tubesheet region, quatrefoil-broached tube hole design with stainless steel TSP material, and supplemental thermal treatment of the row 1 through 9 U-bends following bending. Hydraulic expansion of the tubes in the tubesheet region results in reduced residual stresses compared to mechanical roll

expansion and a more uniform expansion compared to explosively expanded tubes. The broached tube hole condition results in reduced potential for contaminant concentration at TSP intersections by decreasing the crevice area. Supplemental thermal treatment of the row 1 through 9 U-bends following bending is expected to reduce residual stresses to near straight leg region levels. In response to rapid PWSCC initiation in small-radius U-bends in plants with Alloy 600MA tubing, an in situ heat treatment process was developed in the 1980s. Application of this process in plants prior to operation has resulted in a greatly reduced potential for PWSCC initiation. Some of these plants (with MA tubing) have operated for up to 11 EFPYs at hot leg temperatures up to 620°F with no evidence of PWSCC initiation. The supplemental thermal treatment process, performed in the manufacturing phase for the IP2 steam generators, is expected to result in a more effective treatment compared to the in situ heat treatment process.

Condition monitoring and operational assessment programs at IP2 are in place to detect indications of SCC.

5.6.8 References

1. *ASME Boiler and Pressure Vessel Code* Section III, "Rules for the Construction of Nuclear Power Plant Components," 1965 Edition, Summer 1966 Addendum, The American Society of Mechanical Engineers, New York, NY.
2. *ASME Boiler and Pressure Vessel Code*, Section III, "Rules for Construction of Nuclear Vessels," 1989 Edition, The American Society of Mechanical Engineers, New York, NY.
3. NRC Regulatory Guide 1.121, *Bases for Plugging Degraded PWR Steam Generator Tubes (for comment)*, August 1976.
4. EPRI Report TR-104788-R2, *PWR Primary-to-Secondary Leak Guidelines – Revision 2*, EPRI, Palo Alto, CA, 2000.
5. EPRI TR-108501, *Predicted Tube Degradation for Westinghouse Models D5 and F-Type Steam Generators*, Palo Alto, CA, September 1997.

Table 5.6-2

IP2 SPU Evaluation Summary Primary and Secondary Side Components

| Component | Load Condition | Stress Category | Stress (ksi)/ Fatigue - Baseline | Stress (ksi)/ Fatigue - SPU | Allow (ksi)/ Fatigue | Comments |
|--|----------------|----------------------------|----------------------------------|-----------------------------|----------------------|----------------------------|
| Primary Side Components | | | | | | |
| Divider Plate | Normal/Upset | P_m+P_b+Q ⁽¹⁾ | | a,c,e | 69.90 | Plastic analysis performed |
| | | Fatigue | | | 1.00 | |
| Tubesheet & Shell Junction | Normal/Upset | P_m+P_b+Q | | | 58.20 | |
| | | Fatigue | | | 1.00 | |
| Tube-to-Tubesheet weld | Normal/Upset | P_m+P_b+Q | | | 79.80 | |
| | | Fatigue | | | 1.00 | |
| Tubes | Normal/Upset | P_m+P_b+Q | | | 80.00 | |
| | | Fatigue | | | 1.00 | |
| Secondary Side Components⁽²⁾ | | | | | | |
| Main Feedwater Nozzle | Normal/Upset | P_m+P_b+Q | | | 90.00 | |
| | | Fatigue | | | 1.00 | |
| Secondary Manway Bolt | Normal/Upset | P_m+P_b+Q | | | 86.70 | |
| | | Fatigue ⁽³⁾ | | | 1.00 | |
| Secondary Manway Stud | Normal/Upset | P_m+P_b+Q | | | 86.70 | |
| | | Fatigue | | | 1.00 | |
| Steam Nozzle Cross Section | Normal/Upset | P_m+P_b+Q | | | 90.00 | |
| | | Fatigue | | | 1.00 | |
| Insert | Normal/Upset | P_m+P_b+Q | | | 44.10 | |
| | | Fatigue | | | 1.00 | |
| Support Ring | Normal/Upset | P_m+P_b+Q | | | 80.10 | |
| | | Fatigue | | | 1.00 | |

Notes:

- Exceeds 3Sm. Simplified plastic analysis was done in the reference analysis for fatigue evaluation.
- Additional stress due to reduction of pressure is taken to calculate the increase in stress range for secondary side components.
- Bolt fatigue usage is equal to []^{a,c,e}, which is greater than the allowable 1.0. Bolts must be replaced after 34 years of operation, or sooner.

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

Table 5.6-3

Summary of Tube Structural Limits RG 1.121 Analysis

| Location / Wear Scar Length | Parameter | High T _{avg} | Low T _{avg} |
|-------------------------------|-------------------------------------|-----------------------|----------------------|
| Straight Leg | t _{min} (inch) | [|] a,c,e |
| | Structural Limit (%) ⁽¹⁾ | | |
| AVB ⁽²⁾ / 0.5-inch | t _{min} (inch) | [|] a,c,e |
| | Structural Limit (%) ⁽¹⁾ | | |
| FDB / 0.75-inch | t _{min} (inch) | [|] a,c,e |
| | Structural Limit (%) ⁽¹⁾ | | |
| TSP / 1.125-inch | t _{min} (inch) | [|] a,c,e |
| | Structural Limit (%) ⁽¹⁾ | | |

Notes:

1. Structural Limit = $[(t_{nom} - t_{min}) / t_{nom}] \times 100$ percent
 $t_{nom} = 0.050$ in
2. The tube structural limits and minimum thickness specified for the AVB applies only for tube rows 14 and higher. For tube/AVB intersections for tube rows 1 to 13, the structural limits and minimum thickness for the FDB locations are to be used.

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.



a,c,e

Figure 5.6-1
IP2 – Steam Generator MCO at SPU Conditions

5.7 Pressurizer

5.7.1 Structural Analysis

The functions of the pressurizer are to absorb any expansion or contraction of the primary reactor coolant due to changes in temperature and pressure and, in conjunction with the pressure control system components, to keep the Reactor Coolant System (RCS) at the desired pressure. The first function is accomplished by keeping the pressurizer approximately half-full of water and half-full of steam at normal conditions, connecting the pressurizer to the RCS at the hot leg of one of the reactor coolant loops (RCLs), and allowing inflow to, or outflow from the pressurizer as required. The second function is accomplished by keeping the temperature in the pressurizer at the water saturation temperature (T_{sat}) corresponding to the desired pressure. The temperature of the water and steam in the pressurizer can be raised by operating electric heaters at the bottom of the pressurizer and can be lowered by introducing relatively cool spray water into the steam space at the top of the pressurizer.

The components in the lower end of the pressurizer (such as the surge nozzle, lower head/heater well, and support skirt) are affected by pressure and surges through the surge nozzle. The components in the upper end of the pressurizer (such as the spray nozzle, safety and relief nozzle, upper head/upper shell, manway, and instrument nozzle) are affected by pressure, spray flow through the spray nozzle, and temperature differences between the pressurizer steam and the spray water.

The limiting operating conditions of the pressurizer occur when the RCS pressure is high and the RCS hot leg (T_{hot}) and cold leg (T_{cold}) temperatures are low. This maximizes the ΔT that is experienced by the pressurizer. Due to flow out of, and into, the pressurizer during various transients, the surge nozzle alternately sees water at the pressurizer temperature (T_{sat}) and water from the RCS hot leg at T_{hot} . If the RCS pressure is high (which means, correspondingly, that T_{sat} is high) and T_{hot} is low, then the surge nozzle will see maximum thermal gradients (ΔT_{hot} = temperature difference between T_{hot} and the pressurizer (surge nozzle) temperature); and, thus experience the maximum thermal stress. Likewise, the spray nozzle and upper shell temperatures alternate between steam at T_{sat} and spray water, which, for many transients, is at T_{cold} . Thus, if RCS pressure is high (T_{sat} is high) and T_{cold} is low, then the spray nozzle and upper shell will also experience the maximum thermal gradients (ΔT_{cold} = temperature difference between T_{cold} and the pressurizer [spray nozzle] temperature) and thermal stresses.

By evaluating the surge and spray nozzles, all other components are qualified. These evaluations were performed to support the IP2 SPU to address the effect of the SPU on the pressurizer. This evaluation is based on the range of NSSS operating parameters to support a NSSS power level of 3230 MWt (see Table 2.1-2 in Section 2 of this report).

The reactor vessel outlet (T_{hot}) and the reactor vessel/core (T_{cold}) temperatures from Table 2.1-2 define the normal operating temperatures for the surge and spray lines to the pressurizer. The reactor coolant pressure defines the pressurizer normal operating pressure (2250 psia) and saturated temperature (653°F). The minimum values of T_{hot} and T_{cold} from all cases were used in the evaluation of the pressurizer. The NSSS design transients are also applicable to the pressurizer and were considered in the analysis.

The input parameters associated with the IP2 SPU Program were reviewed and compared to the design inputs considered in the current pressurizer stress report. In cases for which revised input parameters are not obviously bounded, pressurizer structural analyses and evaluations were performed. The method of review involves hand calculations using appropriate engineering assessment. Any effects to the existing design basis analysis were evaluated through a comparative analysis of the changes. This method involves a simplified engineering approach, using the existing analyses as basis of evaluation. It uses scaling factors to assess the effect of the changes in the parameters such as the system transients, temperatures, and pressures. New stresses and revised cumulative usage factors are calculated, as applicable, and compared to previous results. The evaluation results show that conformance to the ASME Code allowable limits is maintained. Since the change in the ΔT_{hot} was minimal and bounded by the original design basis calculations, no analyses were necessary for the lower shell and its key components. Only the change in the ΔT_{cold} warranted an analysis of key upper shell components such as the spray nozzle, the safety and relief nozzle, and the upper shell itself.

Conclusions

The analysis shows that the SPU will have a limited effect on the IP2 pressurizer components. Table 5.7-1 compares the fatigue usages calculated for the SPU conditions with those reported from the original design basis. The largest increase was for the spray nozzle for which the fatigue usage increased from []^{a,c,e} to []^{a,c,e}. The fatigue usage for the upper shell decreased significantly due to use of more realistic assumptions on spray effects than were used in the original evaluation. The results for the analyzed components, as shown on Table 5.7-1, envelop all other pressurizer components.

It is concluded that the pressurizer components meet the stress and fatigue analysis requirements of the ASME Code, Section III (Reference 1) for plant operation at the SPU conditions.

5.7.2 References

1. *ASME Boiler and Pressure Vessel Code, Section III, "Rules for Construction of Nuclear Vessels,"* 1965 Edition through Summer 1966 Addenda, The American Society of Mechanical Engineers, New York, NY.

| Table 5.7-1 | | |
|--|-----------------------|------------------------|
| IP2 Pressurizer Component Fatigue Usage Comparison | | |
| Component | Revised Fatigue Usage | Previous Fatigue Usage |
| Spray Nozzle | [] | [] ^{a,c,e} |
| Upper Shell | [] | [] |
| Safety & Relief Nozzle | [] | [] |

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

5.8 Nuclear Steam Supply System Auxiliary Equipment

5.8.1 Introduction

The Nuclear Steam Supply System (NSSS) auxiliary equipment is defined as the equipment contained in the NSSS fluid systems, which are the Reactor Coolant System (RCS), the Chemical and Volume Control System (CVCS), the Residual Heat Removal System (RHRS), the Safety Injection System (SIS), the Component Cooling Water System (CCWS), the Primary Sampling System (PSS), and the Containment Spray System (CSS).

The NSSS auxiliary equipment (auxiliary tanks, heat exchangers, pumps and valves) were reviewed on a system basis for potential effects due to the revised NSSS parameters (the maximum operating temperatures, pressures, and flow rates in Table 2.1-2 in this report) and the revised design transients resulting from the SPU conditions as discussed in Section 3 of this report. The evaluation consisted of a structural and flow capacity review of the component pressure boundaries.

5.8.2 Input Parameters and Assumptions

The NSSS parameters provided in Section 2.1 reflect the effect of the SPU on the NSSS system operating temperatures and pressures. This information was applied where applicable for evaluation of the auxiliary equipment maximum operating temperatures and pressures. Section 3.2 discusses the effect of the SPU on the NSSS auxiliary equipment design transients for the auxiliary tanks, heat exchangers, pumps and valves subject to these transients. Section 3.1 defines the effect of the SPU on the NSSS design transients for the auxiliary system valves subject to these transients.

The evaluation of the NSSS auxiliary equipment was made relative to the technical requirements for the NSSS auxiliary equipment as originally supplied by Westinghouse.

5.8.3 Description of Analyses and Evaluations

The original design parameters, including design temperature, pressure, thermal transients, and flow rates were reviewed for the auxiliary tanks, heat exchangers, pumps and valves. These parameters were compared to those used in the SPU, from Sections 2.1 and 3 of this report, to determine if the design parameters still enveloped those for the SPU.

5.8.3.1 Auxiliary System Tanks

None of the tanks have significant transients identified as part of the original design. From an evaluation of the data and parameters discussed in Sections 2.1 and 3, the operating temperatures and pressures for these vessels remain within the design basis for these tanks, and the SPU transients remain bounded by the original design transients.

5.8.3.2 Auxiliary System Heat Exchangers

The NSSS auxiliary heat exchanger specifications identify the applicable design transients, and the data sheets identify the design temperatures and pressures.

Based on comparison to the NSSS parameters for the SPU, the operating temperature and pressure ranges for these vessels remain bounded by the original design parameters. Section 3 indicates that the original design transients for the auxiliary equipment bound the transients associated with the SPU. The heat exchangers identified in the original design specifications as having transients are the regenerative, letdown, excess letdown, and residual heat removal heat exchangers. All of these temperatures remain bounded by the original design conditions. The RHRS heat exchangers have been structurally evaluated for limiting operating flows during the post LOCA recirculation phase due to various pump alignments. The evaluation indicated the heat exchangers were acceptable for these flows. Therefore, these flows remain valid for the SPU condition as well as the original power condition.

5.8.3.3 Auxiliary System Pumps

The NSSS auxiliary pump specifications identify the applicable design transients, and the data sheets identify the design temperatures and pressures. For the SPU conditions, the operating temperature and pressure ranges for these pumps remain bounded by the original design parameters. Section 3 indicates that the original design transients for the auxiliary equipment bound the transients associated with the SPU.

5.8.3.4 Auxiliary System Valves

The NSSS auxiliary valves specifications identify the applicable design transients, and the data sheets identify the design temperatures and pressures. For the SPU conditions, the operating temperature and pressure ranges for the valves remain bounded by the original design parameters. Section 3 indicates that the original design transients for the auxiliary equipment remain bounded for the transients associated with the SPU.

5.8.4 Acceptance Criteria and Results

If the maximum system operating temperatures, pressures, and flow rates for the SPU Program are bounded by the original system design conditions, then the auxiliary tanks, heat exchangers, pumps and valves are considered to be qualified for the SPU.

If the original design transients bound the revised SPU design transients for the auxiliary tanks, heat exchangers, pumps and valves, then the auxiliary tanks, heat exchangers, pumps, and valves are considered to be qualified for the SPU.

5.8.5 Conclusions

The IP2 auxiliary tanks, heat exchangers, pumps and valves are acceptable for the SPU conditions, since the SPU NSSS parameters are bounded by the original NSSS design parameters (for example, maximum and minimum temperatures) and the original auxiliary equipment design transients.

5.9 NSSS Components Fracture Integrity

5.9.1 Introduction

The Indian Point Unit 2 (IP2) Stretch Power Uprating (SPU) Program involves changes that affect each of the primary NSSS components. This section addresses the effects of the SPU on the fracture integrity of the ferritic Class 1 components, specifically the reactor vessel, steam generators, and pressurizer. These are the components for which non-ductile failure must be considered, according to the requirements of the *American Society of Mechanical Engineers (ASME) Boiler and Pressure Vessel Code, Section III* (Reference 1).

The IP2 reactor vessel was designed to Section III of the 1965 ASME Code. The non-ductile failure requirements were not incorporated into the Code until Appendix G (Reference 2) was added to the 1972 Summer Addenda. The Appendix G analysis for IP2 was completed in November 1974 to comply with the requirements of 10CFR50. That 1974 analysis was used as the basis for the current reactor vessel Appendix G analysis for the SPU Program.

IP2 has the Model 44F steam generator and a Model D Series 84 pressurizer. Generic analyses were used for Appendix G qualification of the steam generator and pressurizer, respectively. These generic analyses were used as the base analyses for these components to assess the effect of the SPU.

5.9.2 Input Parameters and Assumptions

The key input parameters are the stresses in the various components, and the fracture properties of the components. The fracture integrity evaluations for the SPU Program draw on the ASME Code design re-evaluations for the reactor vessel, steam generator components, and pressurizer in Sections 5.1, 5.6, and 5.7, respectively.

The stresses for the baseline reactor vessel analysis were taken from the original IP2 reactor vessel fracture analysis. The original design transients were considered in that reactor vessel fracture analysis, and have been updated to account for the transients discussed in Section 3. The IP2 reactor vessel was previously evaluated as part of the Replacement Steam Generator (RSG) Program. The structural evaluations that were performed are included in an addendum to the reactor vessel stress report and were used in the SPU Program.

The stresses for the baseline steam generator analysis were taken from a typical Model F steam generator stress report. The Model D Series 84 pressurizer analysis was used as the base analysis for the IP2 pressurizer. The stresses obtained from those analyses were adjusted using scale factors previously discussed in earlier sections of this report.

5.9.3 Description of Analyses and Evaluations

5.9.3.1 Methodology

The approach used in the evaluations is a direct application of ASME B&PV Appendix G of Section III (Reference 1). A flaw is postulated, and the crack driving force or stress intensity factor is calculated after adding a safety factor of 2 on the primary stresses. The applied stress intensity factor is then compared with the material fracture toughness, as characterized by the reference stress intensity factor (K_{IR}) toughness curve contained in Appendix G. The following sections detail each of these steps.

5.9.3.2 Stress Intensity Factor Calculations and Postulated Flaw Size

The maximum defect assumed in Appendix G (Reference 1) is a sharp surface defect normal to the direction of the maximum stress. The typical flaw is assumed to be semi-elliptical with an aspect ratio of 1:6 and a depth of one quarter of the vessel wall thickness.

Appendix G (Reference 1) recognizes that some regions cannot be expected to meet the requirements of a one-quarter thickness defect; it states that "smaller defect sizes may be used on an individual case basis if a smaller size of maximum postulated defect can be assured." Welding Research Bulletin 175, *PVRC Recommendations on Toughness Requirements for Ferritic Materials* (Reference 3), provides procedures for considering postulated defect sizes smaller than one quarter of the wall thickness.

The combination of examinations originally required by ASME B&PV Section III (Reference 1) (radiography and surface exams) and the volumetric examination required by Section XI (ultrasonic mapping) are capable of detecting flaws of the magnitude of those assumed for the discontinuity regions for the SPU analyses.

The stress intensity factor, K_I , was calculated for both primary and secondary stress for the limiting transients.

The value of K_I depends on:

- The geometry of the body in which the crack is postulated
- The shape and size of the crack
- The mode and the magnitude of the stress distribution at the crack surface

The general formula of K_I is

$$K_I = M_m \sigma_m + M_b \sigma_b$$

where:

M_m, M_b = the correction factors for membrane and bending stresses, respectively
(depend on the depth and aspect ratio of the crack - see Figure 5.9-1)

σ_m, σ_b = membrane and bending stresses (calculated as if no crack were present)

The general formula is valid for a semi-elliptical surface flaw in both primary and secondary stress conditions.

K_I for primary and secondary stresses should be added to obtain the combined stress intensity factor. Appendix G (Reference 1) requires that a safety factor of 2 be applied to the K_I of primary stresses in normal and upset conditions. A safety factor of 1.5 is to be used for hydrostatic test conditions. Therefore,

$$[K_I]_{\text{combined}} = 2 [K_I]_{\text{primary}} + [K_I]_{\text{secondary}}$$

for normal and upset conditions, and

$$[K_I]_{\text{combined}} = 1.5 [K_I]_{\text{primary}} + [K_I]_{\text{secondary}}$$

for in-service leak and hydrostatic (ISLH) test conditions.

The methodology and the correction factors for calculation of the stress intensity factor for all analyzed regions were taken directly from Appendix G (Reference 1). The expression in Appendix G was developed for a flat plate geometry, but has also been found to be applicable to large diameter vessels. The same expression can be used to model flaws in the nozzle corner region, by setting the plate thickness equal to the nozzle corner throat thickness.

5.9.3.3 Determination of the K_{IR} Curve

The principles of linear elastic fracture mechanics (LEFM) serve as a basis for the evaluation methods of Appendix G of ASME Section III (Reference 1). The central parameter of LEFM is the crack opening mode stress intensity factor K_I . This single parameter defines the elastic stress field in the vicinity of a crack tip. K_I is dependent on the geometry of the body containing the crack, the crack size and shape, and the magnitude and distribution of the stress. A defect will grow unstably whenever K_I exceeds a critical value, K_{IC} , the fracture toughness. The fracture toughness is a material property, dependent on strain rate and temperature. It is also dependent on the metallurgical condition, that is, it changes with microstructure, neutron irradiation, and other metallurgical conditions.

For stress intensity factor rates below 2.5 ksi $\sqrt{\text{in.}}$ /second (the static range), the fracture toughness is indicated by K_{IC} , whereas for higher strain rate (the dynamic range), the critical stress intensity factor is indicated by K_{Id} . A third LEFM parameter, the arrest fracture toughness, K_{Ia} , is the value at which a fast-running crack (unstable propagation) will eventually stop. K_{IC} values are invariably higher than K_{Id} or K_{Ia} values.

The K_{IR} curve essentially represents the lower bound static, dynamic and crack arrest critical K_I values measured as a function of temperature on specimens of SA-533 Grade B Class 1, and SA-508-1, 2, and 3 steel. No available data points for static, dynamic, or arrest tests fall below the curve for K_{IR} .

The temperature scale is defined relative to the reference nil ductility transition temperature, RT_{NDT} . The RT_{NDT} , a nonphysical constant that is related to the brittle-to-ductile fracture transition temperature, is determined by both drop weight tests and Charpy V notch impact tests.

A typical reference fracture toughness curve (K_{IR} versus temperature) is presented in Figure 5.9-2 (Figure G-2110-1 of Reference 1). To facilitate analytical calculations, the equation representing this curve can be expressed as:

$$K_{IR} = 26.78 + 1.233 \exp [0.0145 (T - RT_{NDT} + 160)]$$

Where:

K_{IR} = reference stress intensity factor, ksi $\sqrt{\text{in.}}$

T = temperature at which K_{IR} is permitted, °F

RT_{NDT} = reference nil ductility temperature, °F

A K_{IR} upper shelf of $200 \text{ ksi}\sqrt{\text{in.}}$ has been adopted for unirradiated material, and a shelf of $170 \text{ ksi}\sqrt{\text{in.}}$ has been fixed for irradiated material provided the upper shelf Charpy energy exceeds 50 ft lb. This is a generally accepted industry practice, as shown for example in EPRI Report NP-7195R (Reference 4).

Neutron irradiation adversely affects the toughness properties of the reactor vessel steel. The neutron embrittlement of the steel has been found to be a function of the copper content of the steel for given fluences.

A consequence of a decrease in the toughness properties is a shift in the fracture toughness curve to a higher temperature. Quantitatively, this shift can be assessed by determining the shift to higher temperatures of the initial reference nil ductility temperature RT_{NDT} .

The Nuclear Regulatory Commission (NRC) has also developed copper trend curves for the prediction of RT_{NDT} versus fluence (Reference 5). These curves are presented in Regulatory Guide (RG) 1.99, Revision 2. RG 1.99 curves predict RT_{NDT} shift as a function of nickel content as well as copper content.

The fracture toughness curve, indexed to $T - RT_{NDT}$, therefore, will shift along the abscissa by a value equal to ΔRT_{NDT} for a given level of irradiation and copper content as indicated by the copper trend curves. The RT_{NDT} values at the end of life (EOL) differ sufficiently for the locations, so different reference fracture toughness curves are required.

The fluence drops drastically at a short longitudinal distance beyond the vicinity of the core assemblies as illustrated by Figure 5.9-3. For instance, the nozzles are located more than 30 inches above the top level of the core assembly. The curve in Figure 5.9-3 shows that the fluence is about 0.6 percent of the peak fluence value. This is a typical curve, and not meant to represent IP2 specifically. Thus, the irradiation effects at the nozzle areas become insignificant due to the nozzle locations relative to the core.

The upper head and lower head junctions are located still farther from the core ensuring that there will be no significant irradiation effect at those locations. Consequently, only the K_{IR} curve of the vessel beltline, which is exposed to the maximum irradiation, has been adjusted to account for the shift in RT_{NDT} resulting from irradiation.

The material properties of the reactor vessel are tabulated in Table 5.9-1 along with the initial RT_{NDT} , predicted EOL RT_{NDT} , EOL fluence at the 1/4t location, and cross section thickness of each critical location. For the beltline region, EOL fluence and RT_{PRS} values in Table 5.1-2 of Section 5.1 are used.

5.9.3.4 Acceptance Criteria

The K_I values calculated for the affected regions of the reactor vessel, steam generator and pressurizer were compared with the corresponding material fracture toughness, K_{IR} . Protection against non-ductile failure is then assured if the K_I values were below the K_{IR} values.

The expression used to calculate the stress intensity factor was derived for application to a flaw in a flat plate. An axisymmetrical body provides more constraint than a flat plate does. So, the stress intensities calculated by Appendix G (Reference 1) will be higher than the actual values in the reactor vessel and steam generators.

5.9.4 Analysis and Results

Reactor Vessel—The procedures of Appendix G (Reference 1) were applied to 4 critical locations in the reactor vessel: the bottom head to shell junction, the beltline region, the closure head to upper flange region, and the outlet nozzle to shell region.

The original reactor vessel fracture evaluation was used as the baseline for assessing the effects of the SPU Program. The secondary stresses were adjusted to incorporate the changes described in Section 5.1 for the affected design transients. Since the pressure does not change measurably, the primary stresses are identical to the original analysis results. The reference flaw size was one quarter of the section thickness in all cases, except for the outlet nozzle where a reduced defect size of $1/5t$ was utilized. The justification for a $1/5t$ defect for the nozzle is based on the availability of highly reliable non-destructive inspection techniques that assure capability of detecting such a flaw, because of the greater cross-section thickness at the nozzle-shell juncture, this flaw size is negligibly smaller than a $1/4t$ defect in the other areas of interest.

The combined K_I values for each design transient in Table 5.9-2 are compared with the appropriate EOL K_{IR} curve for the critical locations. Exceptions to this are the plant heatup and cooldown, and ISLH test conditions, which are controlled to be in compliance with Appendix G (Reference 1) margins through the plant Technical Specifications. Table 5.9-2 also shows minimum temperature during each transient for the SPU that is conservatively used for the Appendix G calculation.

The results of the analysis are plotted in Figures 5.9-4 through 5.9-7 for the bottom head to shell junction, the beltline region, the closure head to upper flange region and the outlet nozzle to shell region, respectively. Each transient is represented as a point corresponding to the stress intensity factor and the corresponding minimum temperature during that transient.

The fracture integrity evaluation of the IP2 reactor vessel for the SPU Program is summarized in Table 5.9-3. The results show that the maximum stress intensity factor for the governing transient meets the fracture toughness requirements set by ASME, Section III, Appendix G.

Steam Generator—The procedures of ASME Appendix G (Reference 1) were applied to both primary and secondary side critical components in the steam generators. The Model F steam generator fracture mechanics analysis is applicable to the IP2 steam generators. Since hydrostatic tests are the governing transients for the critical steam generator components, the original Appendix G evaluations still remain valid for the SPU Program. Only normal/upset conditions were affected by the SPU, therefore, only the affected normal/upset conditions were evaluated for the critical steam generator components as part of the SPU Program.

The Model F steam generator stress report was used as the baseline for assessing the effects of the SPU Program. The primary and secondary stresses were adjusted to incorporate the changes described in Section 5.6 for the affected normal/upset transients. The reference flaw size was one fourth of the section thickness in all cases, except for the feedwater-nozzle-to-upper-shell weld, where a flaw size of one seventh of the section thickness was postulated. The temperatures for the affected transients are always at least 432.4°F, so the shell material is always in the upper shelf range of fracture toughness, which is 200 ksi $\sqrt{\text{in.}}$, as for the reactor vessel.

The results in Table 5.9-4 show that the maximum stress intensity factor for the SPU is in all cases less than the fracture toughness, so the steam generators continue to meet the requirements of Appendix G (Reference 1).

Pressurizer—For the pressurizer, the Model D Series 84 pressurizer fracture mechanics analysis was used as the baseline for assessing the effects of the SPU Program. Since the change in the ΔT_{hot} was minimal and bounded by the original design basis, no analyses were necessary for the pressurizer lower shell and its key components. Only the change in the ΔT_{cold} warranted an analysis of key upper shell components such as the spray nozzle, the safety and relief nozzle and the upper shell itself.

To take the change in the ΔT_{cold} into account, a scaling factor was derived as discussed in Section 5.7. The K_I values for the spray nozzle and the safety and relief nozzle were modified for the governing transient using this scaling factor. For the remaining pressurizer components, the existing Appendix G (Reference 1) evaluation remains valid.

The fracture integrity evaluation of the IP2 pressurizer for the SPU Program is summarized in Table 5.9-5. The results show that the maximum stress intensity factors for the governing transients meet the fracture toughness requirements of Appendix G (Reference 1).

5.9.5 Conclusions

The fracture integrity evaluations completed for the SPU Program of the IP2 reactor vessel, steam generators and pressurizer have shown that these components will be in compliance with the fracture integrity design requirements of Appendix G (Reference 1). Such compliance was not originally required by ASME of the reactor vessel because it was manufactured to a code edition that preceded the Summer 1972 Addenda, in which Appendix G first appeared, but IP2 committed to this compliance as a condition for 10CFR50 requirements. The pressurizer and steam generators must comply, and their Appendix G analyses were modified to account for the SPU changes.

5.9.6 References

1. ASME Boiler and Pressure Vessel Code, Section III, *Nuclear Power Plant Components*, (1998 Edition for Appendix G).
2. *ASME Boiler and Pressure Vessel Code*, "Nuclear Vessels," 1965 Edition, The American Society of Mechanical Engineers, New York, NY.
3. Welding Research Bulletin 175, *PVRC Recommendations on Toughness Requirements for Ferritic Materials*, New York, NY, July 1973.
4. EPRI Report NP-7195R, *Flaw Evaluation Procedures: ASME Section XI*, T. U. Marstan, editor, August 1978.
5. NRC Regulatory Guide 1.99, *Radiation Embrittlement of Reactor Vessel Material*, Rev. 2, May 1988.

| Table 5.9-1 | | | | | |
|----------------------------------|---------------------|--------------------------------|--|---|----------------------------------|
| IP2 Reactor Vessel Material Data | | | | | |
| Location | Cu-Wt (%) | Initial RT _{NDT} (°F) | Predicted End of Life RT _{NDT} (°F) | End of Life Fluence at 1/4t ⁽⁴⁾ (n/cm ²) | Cross Section Thickness (inches) |
| Closure Head Flange | N.A. ⁽¹⁾ | 60 ⁽²⁾ | 60 | Negligible | 9.41 |
| Outlet Nozzle | N.A. ⁽¹⁾ | 60 ⁽²⁾ | 60 | Negligible ⁽³⁾ | 10.75 |
| Beltline | 0.25 | 65 | 246 ⁵ | 1.296e19 ⁽⁵⁾ | 8.63 |
| Bottom Head Segment | N.A. ⁽¹⁾ | 15 | 15 | Negligible | 8.63 |

Notes:

1. Not available.
2. Estimated.
3. End of life fluence at 1/5t.
4. End of life fluence based on load factor of 0.8 for 40 years plant operation.
5. For the beltline region, RT_{PTS} and EOL fluence values in Table 5.1-2 of Section 5.1 are used.

| Table 5.9-2 Transient Temperature – IP2 | | | |
|--|------------------------------|--|---|
| No. | Transient | Cold Leg Temperature (for beltline and bottom head) (°F) | Hot Leg Temperature (for outlet nozzle and top head) (°F) |
| 1 | Heatup | | a,c,e |
| 2 | Cooldown | | |
| 3 | Plant Loading | | |
| 4 | Plant Unloading | | |
| 5 | Step Load Increase | | |
| 6 | Step Load Decrease | | |
| 7 | Large Step Load Decrease | | |
| 8 | Loss of Flow | | |
| 9 | Steady State Fluctuations | | |
| 10 | Loss of Load | | |
| 11 | Reactor Trip | | |
| 12 | Cold Hydro | | |
| 13 | Hot Hydro | | |

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

Table 5.9-3
Fracture Integrity Evaluation Summary
IP2 – Reactor Vessel

| Location | Governing Transient | Flaw Depth | Flaw Depth (in.) | K/K _{IR} |
|-------------------------------------|---------------------|---------------------|------------------|----------------------|
| Bottom Head to Shell Junction | Plant loading | 1/4t | 2.16 | [] ^{a,c,e} |
| Beltline Region | Plant loading | 1/4t | 2.16 | [] |
| Closure Head to Upper Flange Region | Loss of flow | 1/4t | 2.35 | [] |
| Outlet Nozzle to Shell Region | Loss of load | 1/5t ⁽¹⁾ | 2.15 | [] |

Note:

- The justification for a 1/5t defect for the nozzle is based on the use of highly reliable non-destructive inspection techniques that assure capability of detecting such a flaw. Per Reference 5, the probability of detecting a flaw 0.5 inch into the base material of nozzle inner radius is greater than 99.9%. A 1/5t flaw has a depth of 2.15 inches.

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

Table 5.9-4

Fracture Integrity Evaluation Summary for SPU Normal/Upset Transients
IP2 – Steam Generators

| Location | Thickness (in.) | Transient | Min. Temp. (°F) | Flaw Depth (in.) | K_I (ksi $\sqrt{\text{in.}}$) | K_{IR} (ksi $\sqrt{\text{in.}}$) |
|--------------------------------------|-----------------|-----------|----------------------|------------------|----------------------------------|-------------------------------------|
| Tubesheet-to-Stub-Barrel Weld | 3.13 | [] | [] ^{a.c.e} | 1.0 | [] ^{a.c.e} | 200 |
| Tubesheet-to-Channel-Head Weld | 5.00 | [] | [] | 1.25 | [] | 200 |
| Feedwater-Nozzle-to-Upper-Shell Weld | 3.52 | [] | [] | 0.50 | [] | 200 |
| Primary Manway Knuckle Region | 12.11 | [] | [] | 3.03 | [] | 200 |
| Lower-Shell-to-Cone Weld | 2.84 | [] | [] | 1.0 | [] | 200 |

Bracketed []^{a.c.e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

Table 5.9-5
Fracture Integrity Evaluation Summary
IP2 – Pressurizer

| Location | Governing Transient | Flaw Depth | K _I /K _{IR} |
|---------------------------------|----------------------|------------|---------------------------------|
| Spray Nozzle (corner region) | [] ^{a,c,e} | 1/4t | [] ^{a,c,e} |
| Safety & Relief Nozzle (corner) | | 0.50 | |
| Upper Shell | | 0.15 | |
| Lower Head/Support Skirt | | 1/4t | |
| Support Lug | | 1/4t | |
| Manway (knuckle region) | | 1/4t | |
| Valve Support Bracket | | 0.13 | |
| Surge Nozzle (corner region) | | 1.42 | |

Bracketed []^{a,c,e} information designates data that is Westinghouse Proprietary, as discussed in Section 1.6 of this report.

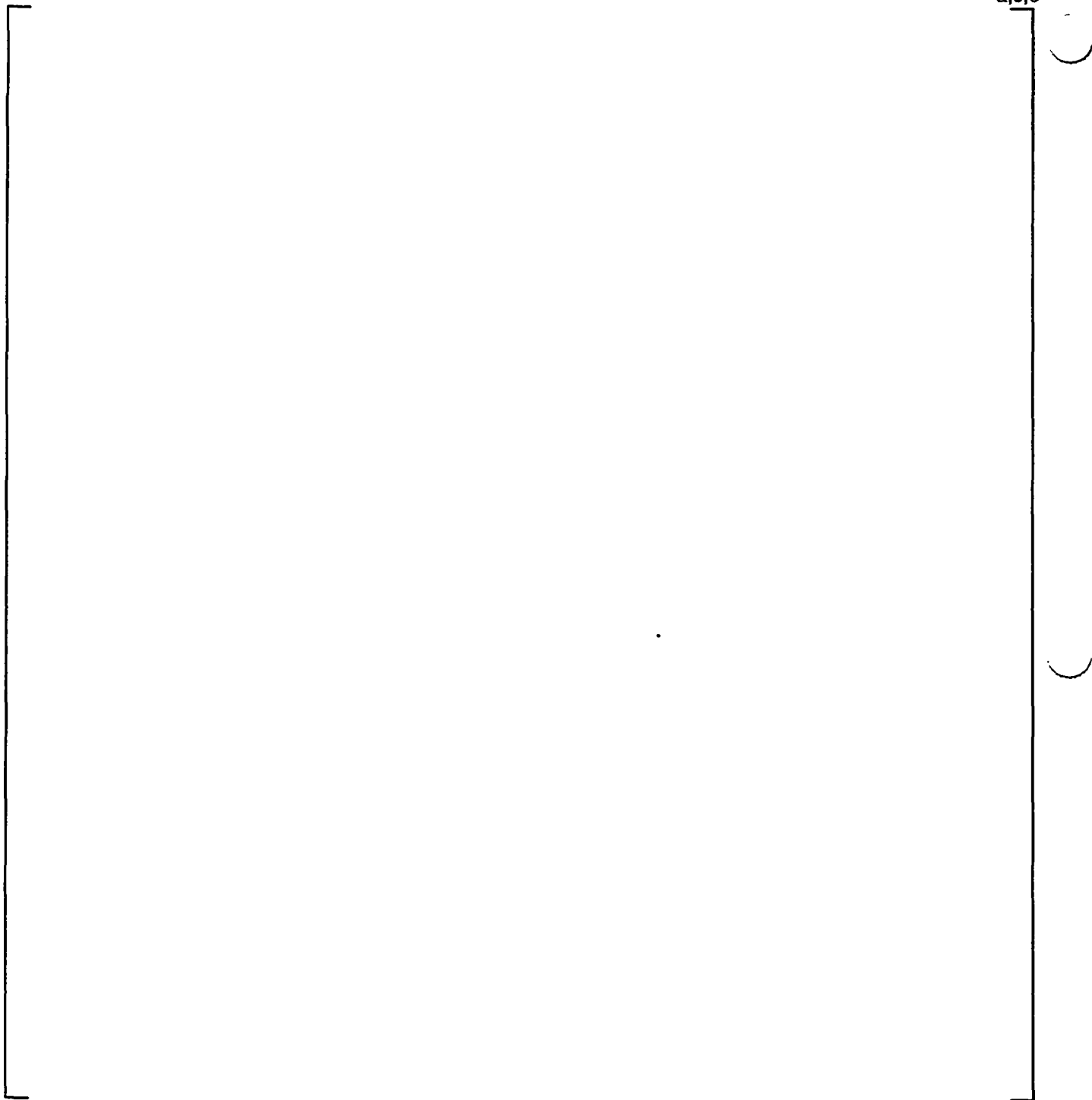


Figure 5.9-1
 M_m and M_b versus $\sqrt{\text{Thickness}}$ Curves

a,c,e

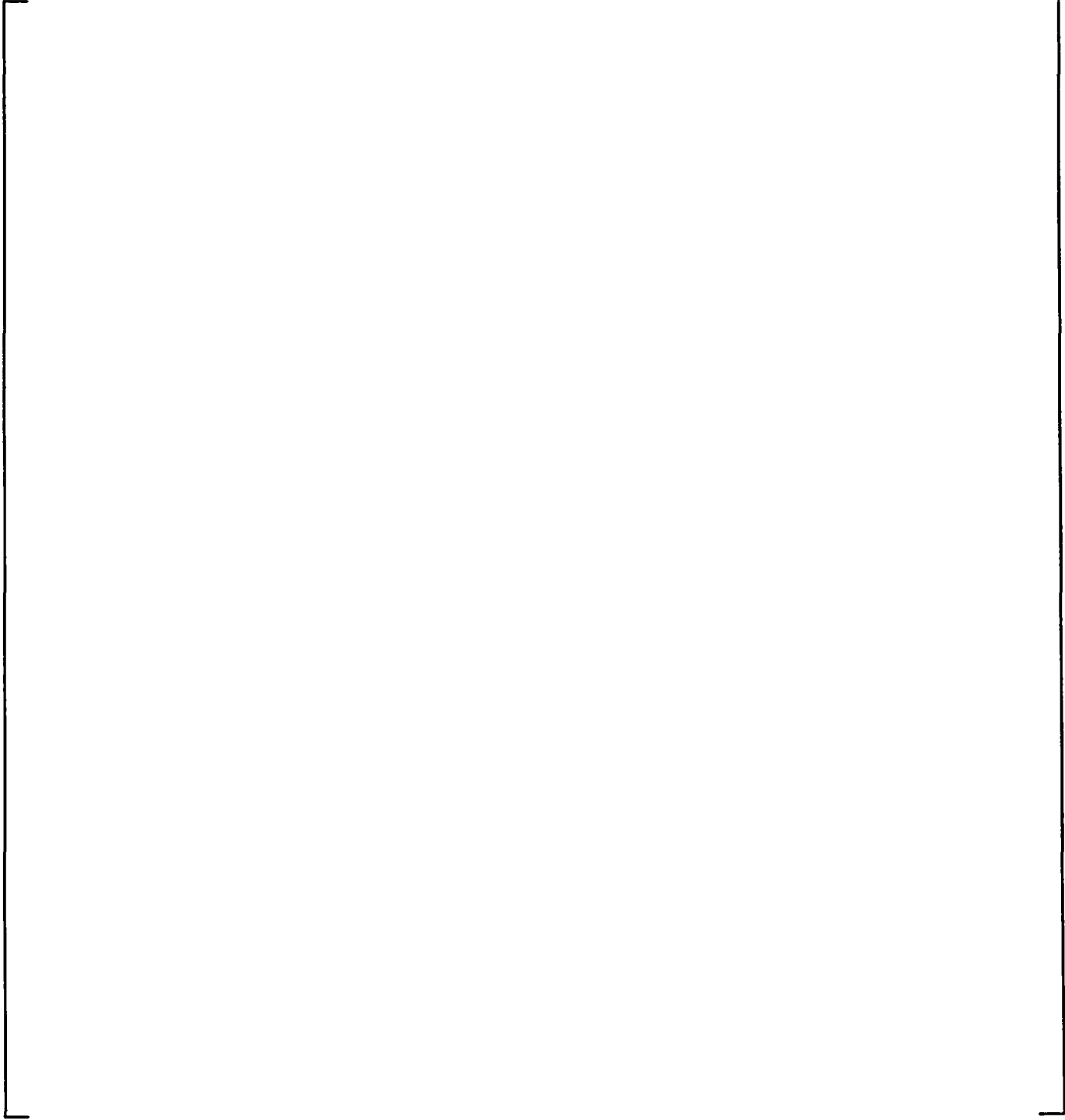


Figure 5.9-2
 K_{IR} Reference Stress Intensity Factor Curve

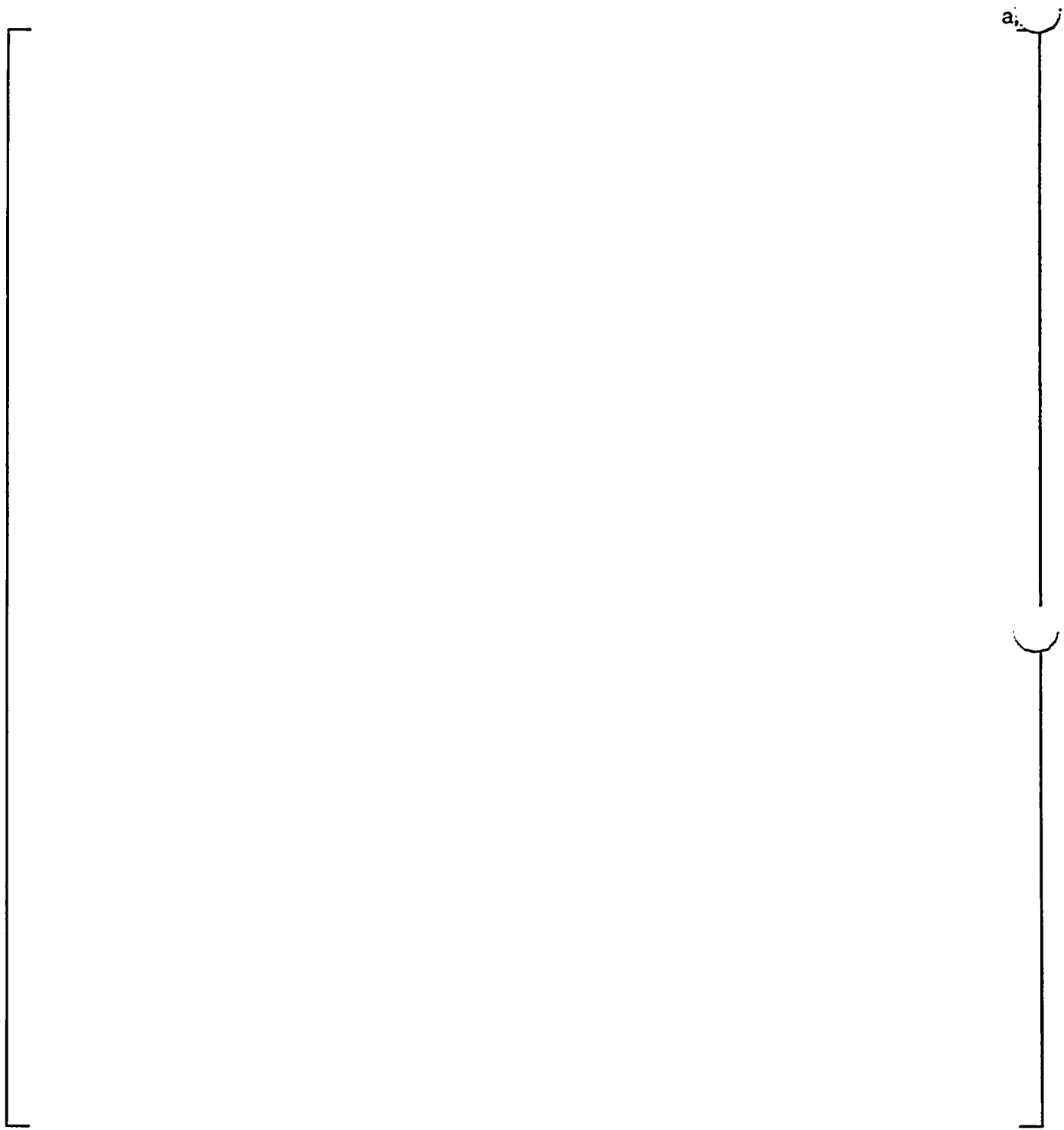


Figure 5.9-3
Longitudinal Distance Versus Multiplying Factor for Peak Fluence



a,c,e



Figure 5.9-4
IP2 Reactor Vessel – Adjusted K_{IR} Curve for Bottom Head to Shell Junction ($RT_{NDT} = +15^{\circ}F$)



Figure 5.9-5
IP2 Reactor Vessel – Adjusted K_{IR} Curve for Beltline Region ($RT_{NDT} = +246^{\circ}F$)



Figure 5.9-6

IP2 Reactor Vessel – Adjusted K_{IR} Curve for Closure Head to Upper Flange Region ($RT_{NDT} = +60^{\circ}F$)

a,c,e

Figure 5.9-7

IP2 Reactor Vessel – Adjusted K_{IR} Curve for Outlet Nozzle to Shell Region ($RT_{NDT} = +60^{\circ}F$)

5.10 RCS Potential Material Degradation Assessment

This section summarizes the evaluations and results of an assessment of the potential materials degradation issues arising from the effects of the Indian Point Unit 2 (IP2) stretch power uprate (SPU) on the performance of primary component materials.

The primary concern from the proposed SPU is the potential effect of changes in the RCS chemistry (impurities) and pH conditions and the SPU service temperatures on the integrity of primary component materials during service. These concerns include general corrosion (wastage) and stress corrosion cracking (SCC) of system materials, fuels corrosion, and primary water stress corrosion cracking (PWSCC) of nickel base alloys. These issues are considered below.

5.10.1 Proposed SPU Service Conditions

A review of the SPU design documents indicates that the following changes in the RCS chemistry and service conditions (Reference 1 and Table 2.1-2) will occur during operations after the SPU implementation:

- The reactor coolant Li/B program is coordinated such that a pH value between 6.9 and 7.4 is maintained initially with a maximum Lithium level of 3.5 ppm. The Lithium level is then decreased gradually during the fuel cycle as the pH value is maintained at a maximum value of less than 7.4 through the end of the fuel cycle.
- The maximum increase in the reactor vessel upper head temperature due to the SPU is estimated at 3.88°F (Table 5.10-1).
- The maximum increase in the hot leg nozzle temperature due to the SPU is estimated at 1.50°F (Table 5.10-1).

5.10.2 Materials Assessment

The effect of the proposed service conditions on the performance of RCS materials is considered below:

Austenitic Stainless Steels

The two degradation mechanisms that are operative in austenitic stainless steels are intergranular stress corrosion cracking (IGSCC) and transgranular stress corrosion cracking (TGSCC). Sensitized microstructure, susceptible materials, and the presence of oxygen are

required for the occurrence of IGSCC, while the introduction of halogens such as chlorides and the presence of oxygen are prerequisites for the occurrence of TGSCC. The chemistry changes resulting from uprating do not involve introduction of any of these contributors so that no effect on material degradation is expected in the stainless steel components as a result of the SPU.

Fuel Cladding Corrosion Effects

An examination of the proposed lithium, boron and pH management program showed that the program adequately meets the proposed EPRI chemistry guidelines (Reference 1). Since these guide lines are specifically designed to prevent fuel cladding corrosion effects such as fuel deposit build-up and Alloy 600 PWSCC, there will be no adverse effect on fuel cladding corrosion. Experience with operating plants as well as with the guidelines provided by EPRI (Reference 1) suggest that increasing initial Li concentrations of up to 3.5 ppm with controlled boron concentrations to maintain pH values between 6.9 to 7.4 has not produced any undesirable material integrity issues and is considered acceptable. IP2 plans to maintain Lithium levels at 3.5 ppm or less. Therefore, there will be no adverse effects from this aspect of the SPU.

Alloy 600/82/182 Components

The most significant factor that influences the PWSCC of Alloy 600/82/182 components is the service temperature. The two most significant Alloy 600/82/182 components that are bounding to the PWSCC susceptibility are the reactor vessel head penetrations (RVHP) and the hot leg nozzle welds. These are considered below.

The Alloy 600 PWSCC susceptibility is a thermally activated process. The PWSCC susceptibility (S) (Reference 2) is given by:

$$S = A(\sigma y k)^4 \exp(-Q/RT)$$

where A is the material constant

$(\sigma y k)^4$ is the stress factor

σy being the yield strength

and k the residual stress factor

Q is the activation energy of the PWSCC process (~50,000 cal/mole)

R the gas constant 1.987

T the temperature in °R

For the current situation, since the only variable due to uprating is the component service temperature, the susceptibility (S) can be expressed as:

$$S = B \exp(-Q/RT), B \text{ being a constant}$$

The change in the PWSCC susceptibility (ΔS) due to a change in the service temperature (ΔT) can be obtained by taking a differential and is given by:

$$\Delta S = B \exp(-Q/RT) (Q/RT^2) \Delta T$$

$$\text{or } \Delta S/S = (Q/RT^2) \Delta T$$

5.10.3 Service Temperature Data

A summary of service temperatures at component locations of interest for various design basis cases is provided in Table 5.10-1. The first two lines of Table 5.10-1 provide the calculated upper head temperature and hot leg nozzle temperatures for the 1990 uprate program conditions and cases (Reference 3) and bound the temperature conditions for the 1.4-percent MUR presented in Table 2.1-1. The last two lines of Table 5.10-1 provide the calculated upper head temperature and hot leg nozzle temperatures for the SPU conditions and cases discussed in Section 2 of this report. See the notes on Table 5.10-1 for details of the cases for each temperature value. The maximum increases in service temperatures (ΔT) at the bounding RVHP and hot leg outlet nozzle weld locations are provided in Table 5.10-1.

5.10.4 Change in the PWSCC Susceptibility of RVHPs

The industry experience over the past decade showed that the PWSCC susceptibility of the Alloy 600/82/182 outer-most circle RVHPs is considered bounding to other Alloy 600 primary component locations due to the presence of high residual stresses and service temperatures at those penetration locations. The RV upper head best-estimate mean fluid maximum service temperature is considered to be the RVHP temperature for the purpose of the current evaluation.

The maximum change in the PWSCC susceptibility value (ΔS) of the highest susceptible (outer circle) penetration was assessed from the maximum change in the RVHP temperature (ΔT_{\max}) due to the SPU. This value was established from the data in Table 5.10-1 to be 3.88°F (3.88°R).

From the equation above:

$$\Delta S/S = (0.08) (\Delta T^{\circ}R)$$

$\Delta S/S$ being the fractional change in the PWSCC susceptibility, and ΔT , the change in the service temperature in units of Rankine.

On this basis, an increase in the PWSCC susceptibility of 31 percent was estimated for the RVHP Penetration as a result of the SPU. The absolute susceptibility of these locations is estimated to be very low ($\sim 10^{-11}$).

5.10.5 Change in the PWSCC Susceptibility of Alloy 82/182 Hot Leg Nozzle Weld

The maximum change in the hot leg nozzle weld PWSCC susceptibility due to the SPU was assessed from the data in Table 5.10-1 to be 1.5°F (1.5°R).

The change in the PWSCC susceptibility value (ΔS) of the highest susceptible hot leg nozzle weld was assessed from the change in the RV outlet nozzle temperature ΔT due to uprating, from the above equation:

$$\Delta S/S = (0.08)(\Delta T)$$

$\Delta S/S$ being the fractional change in the PWSCC susceptibility, and ΔT , the change in the service temperature in units of Rankine.

On this basis, an increase in the PWSCC susceptibility of 12 percent was estimated for the RV Hot Leg Nozzle Weld as a result of the SPU. The absolute susceptibility of these locations is estimated to be very low ($\sim 10^{-11}$).

5.10.6 Conclusions

An assessment of the potential materials degradation issues resulting from the SPU at IP2 concluded that:

- No appreciable material degradation issues were identified with the internal and core support materials due to the SPU at IP2. The lithium concentration will be limited to 3.5 ppm.
- The increase in PWSCC susceptibilities of Alloy 600 RVHP and Alloy 82/182 hot leg nozzle weld locations (31 and 12 percent, respectively) is not considered significant since the absolute susceptibility of these locations is estimated to be very low ($\sim 10^{-11}$).

5.10.7 References

1. EPRI TR-105714 Final Report, *PWR Primary Water Chemistry Guidelines*, Rev. 3, November 1995.
2. *Methodologies to Assess PWSCC Susceptibility of Primary Component Alloy 600 Locations in PWRs*, Proceedings of the 6th International Symposium on Environmental Degradation of Materials, G. V. Rao, NACE, August, 1993.
3. WCAP-11972, *Indian Point Unit 2, NSSS Stretch Rating – 3083.4 MWt Licensing Report*, Rev. 1, Consolidated Edison Company of New York, Inc., March 1989.

Table 5.10-1

**Summary of Change in the Vessel Upper Head and Hot Leg
Nozzle Service Temperatures due to the SPU**

| Core Power Level (MWt) | Location | Lower Bound Temperature (°F) | Upper Bound Temperature (°F) | Maximum Increase in Temperature (ΔT°F)⁸ |
|-------------------------------|-----------------|-------------------------------------|-------------------------------------|---|
| 3071.4 ⁽¹⁾ | RV Upper Head | 571.151 ⁽⁴⁾ | 601.466 ⁽⁵⁾ | -- |
| 3071.4 ⁽¹⁾ | Hot Leg Nozzle | 582.2 ⁽⁴⁾ | 611.7 ⁽⁵⁾ | -- |
| 3216 ⁽²⁾ | RV Upper Head | 575.03 ⁽⁶⁾ | 597.66 ⁽⁷⁾ | 3.88°F |
| 3216 ⁽³⁾ | Hot Leg Nozzle | 583.7 ⁽⁶⁾ | 605.8 ⁽⁷⁾ | 1.50°F |

Notes:

1. Operation at 3071.4 MWt, (Reference 3)
2. SPU calculation for 3216 MWt based on Table 2.1-2 parameters
3. SPU parameters for 3216 MWt from Table 2.1-2
4. Lower bound temperatures at 25% SGTP
5. Upper bound temperatures at 25% SGTP
6. SPU lower bound temperatures at 0% or 10% SGTP
7. SPU upper bound temperatures at 0% or 10% SGTP
8. Decreases in temperature for the SPU relative to the previous evaluation provide a decrease in susceptibility.

6.0 SAFETY ANALYSIS

The Indian Point Unit 2 (IP2) Stretch Power Uprate (SPU) Program included safety analyses for the Updated Final Safety Analysis Report (UFSAR) Revision 17 transients and accidents at power uprate conditions. This section includes the evaluation of initial condition uncertainties at power uprate conditions, which are provided as input to the safety analyses. The results of the safety analyses and setpoint calculations identified whether any changes are required to the reactor trip system (RTS)/engineered safety feature actuation system (ESFAS) setpoints.

In addition to initial condition uncertainties and RTS/ESFAS setpoint changes, the following safety analyses at power uprate conditions are also addressed in this section:

- Loss-of-Coolant Accident (LOCA) Transients
- Non-LOCA Transients
- Steam Generator Tube Rupture (SGTR)
- LOCA Mass and Energy (M&E) Releases
- Main Steamline Break (MSLB) M&E Releases
- LOCA Hydraulic Forces
- Anticipated Transients Without Scram (ATWS)
- Natural Circulation and Cooldown
- Radiological Assessments

The analyses and evaluations presented in this section support operation of IP2 at an uprated core power of 3216 MWt.

6.1 Initial Condition Uncertainties

6.1.1 Introduction

Initial condition uncertainties are conservative steady-state instrumentation measurement uncertainties that are applied to nominal parameter values in order to obtain conservative initial conditions for use in safety analyses. The initial condition uncertainties were recalculated at power uprate conditions for use in the Indian Point Unit 2 (IP2) Stretch Power Uprate (SPU) Program analyses and/or evaluations to assess the acceptability of the safety analyses at power uprate conditions. The initial condition uncertainties for the power uprate conditions were provided as input to the loss-of-coolant accident (LOCA) analysis (Section 6.2), non-LOCA analysis (Section 6.3), steam generator tube rupture (SGTR) analysis (Section 6.4), LOCA mass and energy (M&E) release analysis (Section 6.5), main steamline break (MSLB) M&E release analysis (Section 6.6), LOCA hydraulic forces analysis (Section 6.7), and core thermal-hydraulic design analysis (Section 7.2).

6.1.2 Input Parameters and Assumptions

The uncertainty calculations for the IP2 SPU Program were performed for the uprate operating conditions based on the plant-specific instrumentation and plant calibration and calorimetric procedures.

6.1.3 Description of Analyses and Evaluations

The uncertainty analysis uses the square-root-sum-of-the-squares (SRSS) technique to combine the uncertainty components of an instrument channel in an appropriate combination of those components, or groups of components, which are statistically independent. Those uncertainties that are not independent are arithmetically summed to produce groups that are independent of each other, which can then be statistically combined. The methodology used for the IP2 SPU Program is the same as used for the recently NRC-approved 1.4-percent measurement uncertainty recapture (MUR) (Reference 1).

Initial condition uncertainties were evaluated and recalculated as appropriate for the following six parameters that are explicitly modeled in the IP2 safety analyses:

- Pressurizer Pressure Control - Automatic pressurizer pressure control system (not affected by the SPU)
- RCS T_{avg} Control - Automatic reactor control system

- Reactor Power - Daily calorimetric power measurement (rated thermal power [RTP])
- RCS Total Flow - RCS flow measurements based on a once per fuel cycle calorimetric RCS flow measurement to verify analysis flow assumptions
- Steam Generator Water Level Control - Automatic steam generator water level control system
- Pressurizer Water Level Control - Automatic pressurizer water level control system (not affected by the SPU)

In order to support the start of analyses and/or evaluations for safety analyses early in the IP2 SPU Program, preliminary initial condition uncertainties for power uprate were provided as input to safety analyses and/or evaluations. The initial condition uncertainties for power uprate were then calculated and finalized at a later time during the project, and confirmed to be bounded by the preliminary values. Therefore, although various safety analyses and evaluations may incorporate the preliminary initial condition uncertainties, those allowances are bounding compared to the calculated final values.

6.1.4 Acceptance Criteria and Results

The acceptance criterion for the initial condition uncertainties is that the final calculated values must be bounded by the allowances incorporated in the safety analyses.

The results of the initial condition uncertainty analysis for the IP2 SPU are summarized in Table 6.1-1 along with the allowances incorporated in the safety analyses. Pressurizer Pressure Control and Pressurizer Water Level Control are included for completeness, although these parameters were not affected by the IP2 SPU. This table demonstrates that the safety analyses incorporate uncertainties that are equal to or greater than the final calculated values. The uncertainty calculations for Steam Generator Water Level Control included the resolution of the generic steam generator level uncertainty issues (References 2 through 5), which are unrelated to the power uprate.

6.1.5 Conclusions

Preliminary initial condition uncertainties were determined for the IP2 SPU conditions and were provided as input to the safety analyses and/or evaluations. Final initial condition uncertainties were calculated and confirmed to be bounded by the preliminary initial condition uncertainties.

6.1.6 References

1. WCAP-15904-P, *Power Calorimetric Uncertainty for the 1.4-Percent Uprating of Indian Point Unit 2*, Rev. 1, May 2003.
2. NSAL-02-03, *Steam Generator Mid-deck Plate Pressure Loss Issue*, Rev. 1, April 2002.
3. NSAL-02-04, *Maximum Reliable Indicated Steam Generator Water Level*, Rev. 0, February 2002.
4. NSAL-02-05, *Steam Generator Water Level Control System Uncertainty Issue*, Rev. 1, April 2002.
5. NSAL-03-09, *Steam Generator Water Level Uncertainties*, Rev. 0, September 2003.

| Table 6.1-1 IP2 SPU Summary of Initial Condition Uncertainties | | |
|---|--|---|
| Parameter | Limiting Analysis Initial Condition Uncertainties ⁽¹⁾ | Calculated Final Initial Condition Uncertainties ⁽¹⁾ |
| Pressurizer Pressure Control ⁽²⁾ | | a,c,e |
| RCS T _{avg} Control | | |
| Reactor Power | | |
| RCS Total Flow | | |
| Steam Generator Water Level Control | | |
| Pressurizer Water Level Control High ⁽²⁾ | | |

Notes:

1. A negative bias means the channel indicates lower than actual, and a positive bias means the channel indicates higher than actual.
2. Parameter included, although uncertainty not affected by the SPU.

6.2 Loss-of-Coolant Transients

6.2.1 Large-Break Loss-of-Coolant Accident

A best-estimate large-break loss-of-coolant accident (BELBLOCA) analysis was previously performed at the stretch power uprate (SPU) core power of 3216 MWt with a core power uncertainty of 2 percent and has been the licensing basis analysis for Indian Point 2 (IP2) for several cycles of operation, bounding operation at the lower core power level. This analysis is described in NRC Safety Analysis Report, *Issuance of Amendment for Indian Point Nuclear Generating Unit No. 2* and WCAP-13837 (References 1 and 2).

Subsequent to the analysis but prior to the SPU Program, various plant change evaluations and 10CFR50.46 error-reporting items have resulted in a cumulative peak clad temperature (PCT) of 2176°F. To support the SPU Program, various identified coincidental plant configuration changes were evaluated using varying techniques as discussed in the paragraphs that follow.

6.2.1.1 Plant Change Evaluations

6.2.1.1.1 Reactor Containment Fan Cooler Performance

The maximum reactor containment fan cooler (RCFC) performance was recalculated for the SPU program. The recalculated performance provides lower heat removal rates than the performance previously evaluated. The performance has changed due to closer modeling of the Marlo cooling coils and revisions to the air flow rate for the RCFCs. Lower heat removal rates promote a higher transient containment pressure that is conservative for a BELBLOCA. Therefore, the previous evaluation remains bounding.

6.2.1.1.2 High-Head Safety Injection

The high-head safety injection (HHSI) performance was recalculated for the SPU Program. The HHSI accumulator fill line diversion scenario has been eliminated from consideration in HHSI performance (because of the addition of a separate topping pump), but a 25-gpm allowance for valve leakage has been used. The conservative HHSI performance used in the analysis is lower than the SPU HHSI recalculations. Thus, in withdrawing this scenario for the SPU, there is no BELBLOCA PCT penalty to remove. The resultant HHSI performance after considering all changes implemented for the SPU Program (see Table 6.2-1) exceeds that used in the AOR and thus represents an un-quantified analysis benefit.

6.2.1.1.3 Reactor Coolant System T_{avg} Range and Uncertainty

The Analysis of Record (AOR) and SPU Program values compare as shown in Table 6.2-2.

The AOR included a T_{avg} sensitivity study that demonstrated that higher AOR T_{avg} was limiting. The AOR results thus bound the SPU Program T_{avg} effective range after uncertainty consideration.

6.2.1.1.4 Pressurizer Pressure Uncertainty

The AOR used a range of +/- 60 psi in the statistical PCT study. The SPU Program uncertainty is:

- +/- 25 psi (random), -3 +12 (bias)
- + bias : measured value is higher than actual
- - bias : measured value is lower than actual

The SPU Program pressure range is within the AOR range and, thus, the AOR bounds the SPU pressure range and uncertainty.

6.2.1.1.5 Core Power Uncertainty

The AOR used the standard +/-2 percent uncertainty and Entergy confirmed that the standard +/-2 percent uncertainty should be used for the SPU.

Because of the differences in core power uncertainty, a MONTEC sensitivity study was performed to quantify the effect. The sensitivity resulted in 2°F PCT penalties for both the first and the second reflood-PCT time periods. Subsection 6.2.1.2 of this report provides the cumulative PCT effect.

6.2.1.1.6 Axial Power Distribution

The P_{bot}/P_{mid} axial power distribution range used in the AOR has been previously exceeded with the current range assessed in the Cycle 16 Reload Safety Evaluation (RSE). The SPU Program has widened the range to include the rectangular operating space defined by the following ranges:

- P_{bot} : 0.24 to 0.40
- P_{mid} : 0.30 to 0.43

A MONTEC sensitivity study was performed to quantify this change in axial power distribution. The sensitivity study calculated an 18°F PCT penalty for the first reflood time period and a 1°F PCT penalty for the second reflood time period. These penalties are assessed in addition to the prior assessments assigned for the Cycle 16 RSE. Subsection 6.2.1.2 of this report provides the cumulative PCT effect.

6.2.1.1.7 Peripheral Assembly Power

The AOR models a subset of the peripheral assemblies to be in an explicit WCOBRA/TRAC channel whose rods are at a reduced power. The relative power range of these assemblies as modeled in the AOR is 0.4 to 0.8. The SPU Program value is forecast to be 0.30 to 0.80. For Cycle 15 a similar range was assessed, but the PCT penalty was removed for Cycle 16 operation since the Cycle 16 design was within the AOR range. Using the Cycle 15 evaluation technique, the SPU program PCT penalty is 8°F for both first and second reflood PCT time periods. Subsection 6.2.1.2 of this report provides the cumulative PCT effect.

6.2.1.1.8 Rod Power Census

As noted in WCAP-13837, Chapter 7, Section 3 (Reference 2), the core-wide oxidation (hydrogen generation) study uses a rod power census curve. For the SPU Program, a different census curve has been specified. The core-wide oxidation study was repeated using the same technique with the SPU census curve. The comparison is presented below.

- AOR 0.92 percent (WCAP-13837, Chapter 7, Section 3 [Reference 2])
- SPU 0.94 percent

6.2.1.1.9 Containment Spray

Subsequent to the AOR, a containment spray assessment was performed as part of a larger containment configuration evaluation in year 2000. The SPU Program has confirmed that the injection phase maximum spray performance used in this year 2000 evaluation remains current. Therefore, there is nothing further to evaluate for the SPU Program.

6.2.1.1.10 Pressurizer Level

The full-power pressurizer (PZR) level program for the SPU was determined to be:

$$\begin{aligned} \text{SPU PZR level as a function of } T_{\text{avg}} \text{ (}^\circ\text{F)} \\ &= 37 \text{ percent} && ; T_{\text{avg}} \text{ from 549 to 550} \\ &= 37 + 1.273(T_{\text{avg}} - 550) && ; T_{\text{avg}} > 550 \end{aligned}$$

For the SPU Program T_{avg} range (see subsection 6.2.1.1.3), the AOR PZR level and results show that the proposed PZR level program has a negligible effect on PCT results.

6.2.1.1.11 Thermal Design Flow

As per Table 6.2 of WCAP-13837 (Reference 2), the AOR used a minimum thermal design flow of 80,600 gpm/loop. The SPU Program value is 80,700 gpm/loop (see Section 2 of this report). The AOR value bounds the SPU value and no PCT effect is assessed.

6.2.1.1.12 Steam Generator Tube Plugging

The current BELBLOCA licensing basis steam generator tube plugging (SGTP) level is 20 percent. This bounds the 10-percent value for the SPU program and no PCT effect is assessed.

6.2.1.1.13 Safety Injection Temperature

Table 6.2 of the WCAP-13837 (Reference 2) indicates that the AOR used a maximum value of 120°F for safety injection temperature. The SPU Program value is 110°F. The AOR value bounds the SPU value and no PCT effect is assessed.

6.2.1.1.14 Accumulator Gas Cover Pressure

Table 6.2-3 of the WCAP-13837 (Reference 2) indicates that the AOR used a range of 587 to 685 psig for accumulator gas cover pressure. The SPU Program range is 598 to 685 psig as provided by Entergy. The AOR range bounds the SPU range and no PCT effect is assessed.

6.2.1.1.15 15 x 15 Upgraded Fuel

Upgraded fuel is being evaluated for implementation with the SPU. WCAP-13837, *Best Estimate Analysis of the Large Break Loss of Coolant Accident for Indian Point Unit 2 Nuclear Plant*, Revision 1 provides an analysis that bounds both optimized fuel assembly (OFA) and Vantage+ fuels (Table 6-2 of Reference 2). As a first step, an evaluation was performed to rebaseline the PCT to current Vantage+ fuel. Included in the rebaseline evaluation are steady-state maximum average fuel temperatures calculated by the PAD 4.0 code (Reference 3). The rebaseline evaluation resulted in a benefit of 89°F and 95°F for first and second reflood PCT periods, respectively.

The second step in the evaluation was to establish the penalty for the hydraulic mismatch during transition to the upgraded fuel. Penalties of 8 and 26°F were assigned to the first and second reflood PCT periods, respectively.

A third evaluation was made for the full core of upgraded fuel. For the first reflood PCT period a penalty of 14°F was assigned relative to the transition core. For the second reflood PCT period, a benefit of 17°F was assigned relative to the transition core.

Subsection 6.2.1.2 of this report provides the cumulative PCT effect.

6.2.1.2 Results and Conclusions

Table 6.2-3 presents a summary of the PCT changes accrued as a result of the evaluations discussed in subsection 6.2.1.1 of this report. When added to the prior cumulative PCT, the PCT remains below the 2200°F 10CFR50.46 acceptance criteria for both the transition core and for the 15 x 15 upgrade fuel full core.

6.2.2 Small-Break Loss-of-Coolant Accident

6.2.2.1 Introduction

A small-break loss-of-coolant accident (SBLOCA) analysis was performed to support the Stretch Power Uprate (SPU) Program for IP2. The analysis was performed to demonstrate conformance with the 10CFR50.46 requirements (Reference 4) for the conditions associated with the SPU and to explicitly include modeling of items for which the AOR had PCT assessments applied. The approved Westinghouse SBLOCA Evaluation Model (EM) was used for this analysis (References 5 and 6). The SBLOCA EM update that has been approved by the Nuclear Regulatory Commission (NRC) (References 5 and 6) has been used in this analysis, including the COSI condensation model and safety injection (SI) in the broken loop (Reference 7).

6.2.2.2 Input Assumptions and Initial Conditions

6.2.2.2.1 Assumptions

All of the assumptions required by Appendix K to 10CFR50 (Reference 8) have been made in the IP2 SBLOCA analysis. This analysis returns to the assumption of a 2-percent power uncertainty by assuming 102 percent of full power as the initiating condition for the SBLOCA. Other Appendix K assumptions include, but are not limited to, all peaking factors simultaneously at their most limiting values, Baker-Just zirconium-water reaction rate, 120 percent of 1971

American Nuclear Society (ANS) infinite life decay heat, and Moody break flow during periods when two-phase flow is calculated to occur at the break.

Among the major assumptions inherent in the Westinghouse Appendix K SBLOCA EM are:

- Break area is $<1 \text{ ft}^2$.
- SBLOCA initiates at hot full power (HFP) (Mode 1).
- All rod cluster control assemblies (RCCAs), except the single most reactive, insert following reactor trip.
- Loss-of-offsite power (LOOP) assumed at reactor trip time results in the following assumptions:
 - Loss of 1 emergency diesel generator (EDG) and subsequent loss of 1 train of pumped Emergency Core Cooling System (ECCS)
 - Reactor coolant pump (RCP) trip and coastdown
 - Main steam line isolation (no steam dump capability)
- Standard four-loop ECCS spilling assumptions

A spectrum of 3 break sizes, including diameters of 2, 3, and 4 inches, was analyzed.

6.2.2.3 Description of Methodology/Analysis

6.2.2.3.1 Description of SBLOCA Engineering Methodology and Codes

The small-break analysis was performed with the Westinghouse ECCS EM using NOTRUMP (References 5 and 6), including changes to the model and methodology as described in Reference 7. The NOTRUMP EM includes the following computer codes:

NOTRUMP: Thermal-hydraulic response of Reactor Coolant System (RCS) during transient

SBLOCTA: Fuel rod/cladding heat-up during transient

6.2.2.3.2 Description of Analysis Performed for SBLOCA

The methodology first calculated the system thermal-hydraulic response to the SBLOCA event using the NOTRUMP code. These results are then analyzed for their effect on the hot rod heat up using the SBLOCTA code to demonstrate that the PCT, cladding oxidation, and hydrogen generation are below their limiting values as defined by 10CFR50.46 (Reference 4).

6.2.2.3.3 Limiting SBLOCA Sequence

The analysis consists of a break spectrum using the approved methodology as documented in References 5 and 6 and extended in Reference 7. For the IP2 SBLOCA analysis, a three-break spectrum (2-, 3-, and 4-inch) has been analyzed to confirm that the 3-inch break remains limiting. The results are presented in Tables 6.2-4 and 6.2-5.

6.2.2.4 Design Basis Acceptance Criteria

The criteria for acceptability for LOCAs are found in 10CFR50.46(b) (Reference 4) and are quoted below:

1. **Peak cladding temperature:** The calculated maximum fuel element cladding temperature shall not exceed 2200°F.
2. **Maximum cladding oxidation:** The calculated total oxidation of the cladding shall nowhere exceed 0.17 times the total cladding thickness before oxidation.
3. **Maximum hydrogen generation:** The calculated total amount of hydrogen generated from the chemical reaction of the cladding with water or steam shall not exceed 0.01 times the hypothetical amount that would be generated if all of the metal in the cladding cylinders surrounding the fuel, excluding the cladding surrounding the plenum volume, were to react.
4. **Coolable geometry:** Calculated changes in core geometry shall be such that the core remains amenable to cooling.
5. **Long-term cooling:** After any calculated successful initial operation of the ECCS, the calculated core temperature shall be maintained at an acceptably low value and decay heat shall be removed for the extended period of time required by the long-lived radioactivity remaining in the core.

6.2.2.5 Results and Conclusions

6.2.2.5.1 Description of Limiting 3-Inch Break Case

For the limiting 3-inch break, the primary side pressure begins a rapid drop at the time of break initiation (Figure 6.2-1). A reactor trip signal is generated at 18.2 seconds, followed by a SI signal at 26.4 seconds. This primary side depressurization is checked when the primary side saturation temperature reaches the secondary side saturation temperature, since the steam generators provide the predominant energy release path during this portion of the transient. When the loop seal in the broken loop clears at approximately 560 seconds, a vapor vent path is created between the top of the core and the break in the cold leg.

At break initiation, the core mixture level (Figure 6.2-2) drops rapidly until it reaches the elevation at the top of the hot legs. The rate of core level draining is then slowed as vapor is now allowed to enter the hot legs or the inner vessel due to the loop seal clearing. When the core mixture level decreases below the bottom of the hot legs, the mixture level again decreases until loop seal clearing occurs (Figure 6.2-2). After loop seal clearing, the core and downcomer come into a manometric balance as the downcomer level falls in response to the adjacent cold legs draining.

The core mixture level continues to decrease until the top of the core uncovers at 629 seconds, leading to the start of clad heat up. As illustrated in Figure 6.2-3, the SI flow rate continues to increase as the RCS pressure decreases (Figure 6.2-1). The SI replenishes the core level, which results in a reversal in the clad heat up transient. The PCT of 1028°F occurs at 1308 seconds (Figure 6.2-4), followed by a steady increase in the core mixture level until the core recovers at 1924 seconds (Figure 6.2-2). The accumulators inject at 1689 seconds. The transient core exit steam flow has been presented in Figure 6.2-5. The results of the 3-inch break case are presented in Tables 6.2-4 and 6.2-5.

6.2.2.5.2 Non-Limiting Results

The results of the 2- and 4-inch break cases are presented in Tables 6.2-4 and 6.2-5. Figures 6.2-6 through 6.2-11 pertain to the 2-inch and 4-inch break cases. The figures provided for the non-limiting cases are:

- Figure 6.2-6 – 2-Inch Break, Pressurizer Pressure
- Figure 6.2-7 – 2-Inch Break, Core Mixture Level
- Figure 6.2-8 – 2-Inch Break, PCT at PCT Elevation (10.75 ft)
- Figure 6.2-9 – 4-Inch Break, Pressurizer Pressure
- Figure 6.2-10 – 4-Inch Break, Core Mixture Level
- Figure 6.2-11 – 4-Inch Break, PCT at PCT Elevation (11.0 ft)

6.2.2.5.3 10CFR50.46 PCT Report Item Incorporation

As a result of this analysis, all items from the IP2 10CFR50.46 (Reference 4) PCT report are eliminated. This was accomplished by using the latest version of the NOTRUMP EM codes and incorporating each of the other miscellaneous items into the analysis.

6.2.2.5.4 Maximum Local and Core-Wide Oxidation

All cases meet the 10CFR50.46 requirements of maximum local and core-wide oxidation. The local oxidation of the cladding does not exceed 17 percent, and the calculated total amount of hydrogen generated from the chemical reaction of the cladding with water or steam does not exceed 1 percent.

6.2.2.5.5 Conclusions

The results of the analysis show that the acceptance criteria discussed in subsection 6.2.2.4 of this document for the SBLOCA have been met. The limiting PCT for IP2 will be reported as 1028°F, which occurs for the 3-inch break case. Local oxidation of the cladding is less than 17 percent, and the core-wide oxidation is less than 1.0 percent. Results for the 3-inch limiting break case are shown in Figures 6.2-1 through 6.2-5.

6.2.3 Post-LOCA Subcriticality and Long-Term Core Cooling

6.2.3.1 Introduction

The post-LOCA subcriticality calculations support evaluations that demonstrate that the core will remain subcritical upon entering the sump recirculation phase of Emergency Core Cooling System (ECCS) injection. During the sump recirculation phase, safety injection (SI) flow is drawn from the containment sump following switchover from the refueling water storage tank (RWST). To show that the sump water has sufficient boron concentration, the sump mixed mean boron concentration is calculated. The mixed mean boron concentration of the sump water is a function of the various water and boron contributors to the sump prior to start of sump recirculation. The boron concentration of the sump water must be sufficient to keep the core subcritical. The sump-mixed mean boron concentration calculations are used to develop a post-LOCA subcriticality boron limit curve that is confirmed on a cycle-specific basis as part of the Westinghouse Reload Safety Evaluation Methodology (Reference 9). Long-term core cooling also requires adequate ECCS flow to provide core cooling during the cold leg recirculation period.

6.2.3.2 Input Parameters and Assumptions

The sump-mixed mean-boron concentration calculation model is based on the following assumptions:

- Boron is mixed uniformly in the sump. The post-LOCA sump inventory is made up of constituents that are equally likely to return to the containment sump; that is, selective holdup in containment is neglected.
- The calculation of the sump-mixed mean boron concentration assumes minimum mass and minimum boron concentrations for significant boron sources and maximum mass and minimum boron concentration for significant dilution sources.
- The sump-mixed mean boron concentration is calculated as a function of the pre-trip RCS conditions.

The Westinghouse licensing position for satisfying the requirements of 10CFR50.46 Paragraph (b), Item (5), "Long-Term Cooling," is documented in WCAP-8339 (Reference 10). The Westinghouse position is that the core will remain subcritical post-LOCA by borated water from various injected ECCS water sources. To provide subcriticality when entering sump recirculation, the borated ECCS water provided by the accumulators and RWST must have a sufficiently high boron concentration that, when mixed with other sources of borated and non-borated water, the core will remain subcritical. Consistent with the position in WCAP-8339 (Reference 10), control rods are assumed to be withdrawn from the core.

Long-term core cooling also requires adequate ECCS flow to provide core cooling. For IP2, the confirmation of adequate ECCS flow during the cold leg recirculation period is based on the following assumptions:

- The current SBLOCA analysis methodology explicitly models ECCS flow enthalpy changes during the switchover from cold leg injection to cold leg recirculation.
- The long-term core cooling methodology assumes that large-break ECCS flows are not adversely affected by the switchover from cold leg injection to cold leg recirculation.

6.2.3.3 Description of Analyses and Evaluations

Although core power level is not a direct input in the sump-mixed mean boron concentration calculation, the T_{avg} range associated with power uprate conditions will have a minor effect on the RCS fluid masses used in the calculation. Furthermore, all of the inputs used in the calculation were reviewed to confirm consistency with the Technical Specifications and consistency with the assumptions used in the other LOCA analyses being performed for the uprate.

A post-LOCA subcriticality boron limit curve was developed for the uprated conditions using the input parameters and assumptions described in the previous section. The minimum RWST boron concentration for the uprating calculations increased from 2000 to 2400 ppm.

6.2.3.4 Acceptance Criteria and Results

There are no specific acceptance criteria in generating the post-LOCA sump boron concentration curve. However, the resulting curve, which is calculated as a function of the initial RCS peak xenon boron concentration, is included in the Reload Safety Analysis Checklist and is verified for each reload cycle to confirm that adequate boron exists to maintain subcriticality in the long-term post-LOCA.

The post-LOCA sump boron concentration was calculated for RCS boron concentrations of 0 and 1500 ppm, assuming the pre-trip RCS boron concentration for peak xenon concentrations to be 100 ppm lower than the equilibrium case. Figure 6.2-12 shows the post-LOCA sump boron concentration curve.

With respect to long-term core cooling, the SBLOCA analysis discussed in subsection 6.2.2 of this report modeled the ECCS flow enthalpy change during the switchover from cold leg injection to cold leg recirculation. For an LBLOCA, the minimum flows provided by the ECCS for switchover from cold leg injection to cold leg recirculation are adequate to provide long-term core cooling.

6.2.3.5 Conclusions

A post-LOCA subcriticality boron limit curve was developed for the uprated conditions. This curve will be used to evaluate the fuel loading arrangement on a cycle-by-cycle basis during the fuel reload process. Provided that the maximum critical boron concentration remains below the post-LOCA sump boron concentration limit curve (for all rods out, no xenon, 68 to 212°F), the core will remain subcritical post-LOCA, and decay heat can be removed for the extended period

required by the remaining long-lived radioactivity. ECCS flow during the cold leg recirculation period is adequate to provide long-term core cooling.

6.2.4 Hot Leg Switchover

6.2.4.1 Introduction

A post-LOCA hot leg switchover (HLSO) time is calculated to support Emergency Operating Procedures (EOPs) that require a realignment of the recirculation safety injection (SI) flowpath from the cold legs to the hot legs. This realignment to the hot legs precludes boron precipitation in the reactor vessel following a large-break LOCA (LBLOCA). At issue are cold leg breaks where injected SI water boils off due to decay heat, leaving behind boric acid. The concern is the possibility that eventually the boric acid solution in the vessel may reach the boron precipitation point. The Westinghouse ECCS evaluation model relies on the preclusion of boron precipitation as one criteria for ensuring core coolable geometry.

6.2.4.2 Input Parameters and Assumptions

The IP2 HLSO calculation model is based on the following assumptions:

- A boric acid concentration level is computed over time for a core-region mixing volume. Other than the steam exiting through the hot legs and the corresponding makeup SI entering through the lower plenum, there are no other assumed flow paths in or out of the mixing volume. All boric acid entering this mixing volume remains in this mixing volume prior to initiation of hot leg recirculation. The water/boric acid solution is well mixed in the mixing volume region. The water/boric acid solution in the vessel is assumed to be at atmospheric conditions, at a temperature of 212°F. The collapsed mixture level of the core/upper plenum region is at the bottom of the hot leg flow area at the reactor vessel. This level is the top of the mixing volume. The bottom of the mixing volume is at the level of the top of the lower core plate. The lower plenum volume and barrel baffle region volume are not included in the mixing volume.
- The boric acid concentration limit is the experimentally determined boric acid saturation concentration with a 4 weight-percent uncertainty factor. There is no allowance for increase in boric acid solubility due to other solutes such as TSP. The calculation does not include any elevation of boiling temperature due to concentration of boric acid in the core or due to backpressure from containment.

- The decay heat generation rate is based on the 1971 ANS Standard for infinite operating time plus 20-percent margin. The decay heat generation includes a core power multiplier to address instrumentation uncertainty as identified by Section I.A of Appendix K (Reference 8).
- The boron concentration of the make-up SI water during recirculation is a calculated sump-mixed mean boron concentration. The calculation of the sump-mixed mean boron concentration assumes maximum mass and maximum boron concentrations for significant boron sources and minimum mass and maximum boron concentrations for significant dilution sources.
- Once realigned to hot leg recirculation, the minimum recirculation flows for the hot legs, cold legs, or simultaneous hot and cold leg recirculation are confirmed to be sufficient to provide core cooling and preclude boron precipitation.

The methodology described above is consistent with, or otherwise conservative with respect to, the methodology described in CLC-NS-309 (Reference 11).

6.2.4.3 Description of Analyses and Evaluations

The major inputs to the HLSO time calculation include the core power assumptions and boron concentrations and water volume/masses for significant contributors to the containment sump. With the increase in boron concentrations of the RCS, refueling water storage tank (RWST), and accumulators, and since the increase in core power to 3216 MWt affects decay heat, the HLSO time and hot leg recirculation minimum required flows must be recalculated. An increase in core power and boron concentrations will reduce the HLSO time and increase the hot leg recirculation minimum required flows.

For the uprate program, a new HLSO time was calculated using the input parameters and assumptions described in the previous section, except that decay heat was based on the 1971 ANS standard for infinite operation with 20-percent margin. All inputs to the calculation were reviewed and confirmed to be appropriate for plant operation at the uprate conditions. The uprating calculations used the uprated core power of 3216 MWt with a 1.02 calorimetric uncertainty multiplier to address instrumentation uncertainty. Also, the uprating calculations used an increase in boron concentration from 2000 to 2600 ppm for the RWST and accumulators, and an increase in boron concentration from 2000 to 2400 ppm for the RCS.

A revised set of hot leg recirculation minimum required flows were calculated at the uprate conditions and new HLSO time.

6.2.4.4 Acceptance Criteria and Results

There are no specific acceptance criteria on the new HLSO time for the power uprate conditions as long as the *Updated Final Safety Analysis Report* (UFSAR) and EOPs are revised appropriately. The available flows at HLSO time are acceptable if they are shown to be sufficient to provide core cooling.

At the uprated conditions, a new HLSO time of 6.76 hours was calculated. The time of 6.76 hours was rounded down to 6.5 hours for added conservatism. The minimum hot leg recirculation flows at an HLSO time of 6.5 hours and a power level of 3216 MWt are sufficient to preclude boron from precipitating in the vessel and to ensure adequate core cooling is maintained.

6.2.4.5 Conclusions

An HLSO time of 6.5 hours will preclude boron precipitation for post-LOCA scenarios for the uprated conditions. The available ECCS flows at HLSO were shown to be sufficient to provide core cooling and preclude boron from precipitating in the core.

6.2.5 References

1. NRC Safety Evaluation Report, *Issuance of Amendment for Indian Point Nuclear Generating Unit No. 2*, (TAC NO. M96370), J.F. Harold (NRR) to S. E. Quinn (Con Ed), March 31, 1997.
2. WCAP-13837 (Nonproprietary), *Best Estimate Analysis of the Large Break Loss of Coolant Accident for Indian Point Unit 2 Nuclear Plant*, Rev. 1, S. B. Nguyen, and M. Y. Young, December 1996.
3. WCAP-15063-P-A, *Westinghouse Improved Performance Analysis and Design Model (PAD 4.0)*, Foster, Sidener, and Slagle, Rev. 1 with Errata, July 2000.
4. 10CFR50.46, *Acceptance Criteria for Emergency Core Cooling Systems for Light Water Cooled Nuclear Power Reactors*.
5. WCAP-10054-P-A, *Westinghouse Small Break ECCS Evaluation Model Using the NOTRUMP Code*, N. Lee, et al., August 1985.
6. WCAP-10079-P-A, *Notrump A Nodal Transient Small Break and General Network Code*, August 1985.

7. *WCAP-10054-P-A, Addendum to the Westinghouse Small Break ECCS Evaluation Model Using the NOTRUMP Code: Safety Injection into the Broken Loop and COSI Condensation Model, Addendum 2, Rev. 1, C. M. Thompson, et al., July 1997.*
8. *10CFR50 Appendix K, ECCS Evaluation Models.*
9. *WCAP-9272-P-A (Proprietary), Westinghouse Reload Safety Evaluation Methodology, F. M. Bordelon et al., July 1985.*
10. *WCAP-8339, Westinghouse ECCS System Evaluation Model - Summary, June 1974.*
11. *Letter CLC-NS-309, C. L. Caso (Westinghouse) to T. M. Novak (NRC), April 1, 1975.*

| Table 6.2-1 BELBLOCA Minimum HHSI Injection Data | | |
|---|---------|---------|
| | AOR | SPU |
| RCS Pressure (psig) | lbm/sec | lbm/sec |
| 0 | 90.78 | 103.4 |
| 100 | 85.51 | 98.4 |

| Table 6.2-2 Comparison of AOR and SPU Values | | |
|---|---------------------------------|---------------------------|
| | BELBLOCA T _{avg} Range | |
| RCS T _{avg} | AOR | Uprate |
| Nominal | 549-583 | 549-572 |
| Uncertainty | +/- 4 | +/- 3.6 Random -3 Bias |

**Table 6.2-3
BELBLOCA SPU Program PCT Summary**

| Item | Note | PCT (°F) | |
|--|------------------------|-------------------------|-------------------------|
| | | 1 st Reflood | 2 nd Reflood |
| Starting PCT | Reference 2 | 2176 | 2145 |
| Core Power Uncertainty | Subsection 6.2.1.1.5 | +2 | +2 |
| P _{bot} /P _{mid} Operating Space | Subsection 6.2.1.1.6 | +18 | +1 |
| Peripheral Assembly Minimum Power | Subsection 6.2.1.1.7 | +8 | +8 |
| | | | |
| Rebaseline (w/ PAD4.0) | Subsection 6.2.1.1.15 | -89 | -95 |
| Upgrade Fuel Transition Cycles | Subsection 6.2.1.1.15 | +8 | +26 |
| Upgrade Fuel Full Core Cycles | Subsection 6.2.1.1.15 | +14 | -17 |
| | | | |
| Revised Total | During fuel transition | 2123 | 2087 |
| Revised Total | After fuel transition | 2137 | 2070 |

| Table 6.2-4 | | | |
|----------------------------------|---------------|---------------|---------------|
| NOTRUMP Transient Results | | | |
| Event Time (sec) | 2-Inch | 3-Inch | 4-Inch |
| Break Initiation | 0.0 | 0.0 | 0.0 |
| Reactor Trip Signal | 43.3 | 18.2 | 10.4 |
| S-Signal | 61.0 | 26.4 | 14.9 |
| SI Begins | 86.0 | 51.4 | 39.9 |
| Loop Seal Clearing* | 1270. | 560. | 286. |
| Core Uncovery | 1711. | 629. | 692. |
| Accumulator Injection Begins | N/A | 1689. | 850. |
| Core Recovery | 3854. | 1924. | 1170. |

* Loop seal clearing is defined as break vapor flow >1 lb/s.

| Table 6.2-5 | | | |
|---|---------------|---------------|---------------|
| Beginning-of-Life (BOL) Rod Heatup Results | | | |
| | 2-Inch | 3-Inch | 4-Inch |
| Time-in-Life | BOL | BOL | BOL |
| PCT (°F) | 938 | 1028 | 878 |
| PCT Time (s) | 1967 | 1308 | 955 |
| PCT Elevation (ft) | 10.75 | 11.0 | 11.0 |
| HR Burst Time (s) | N/A | N/A | N/A |
| HR Burst Elevation (ft) | N/A | N/A | N/A |
| Max. Local ZrO ₂ (%) | 0.01 | 0.02 | <1.0 |
| Max. Local ZrO ₂ Elev (ft) | 11.25 | 11.0 | 11.25 |
| Hot Rod Axial Avg. ZrO ₂ (%) | <1.0 | <1.0 | <1.0 |

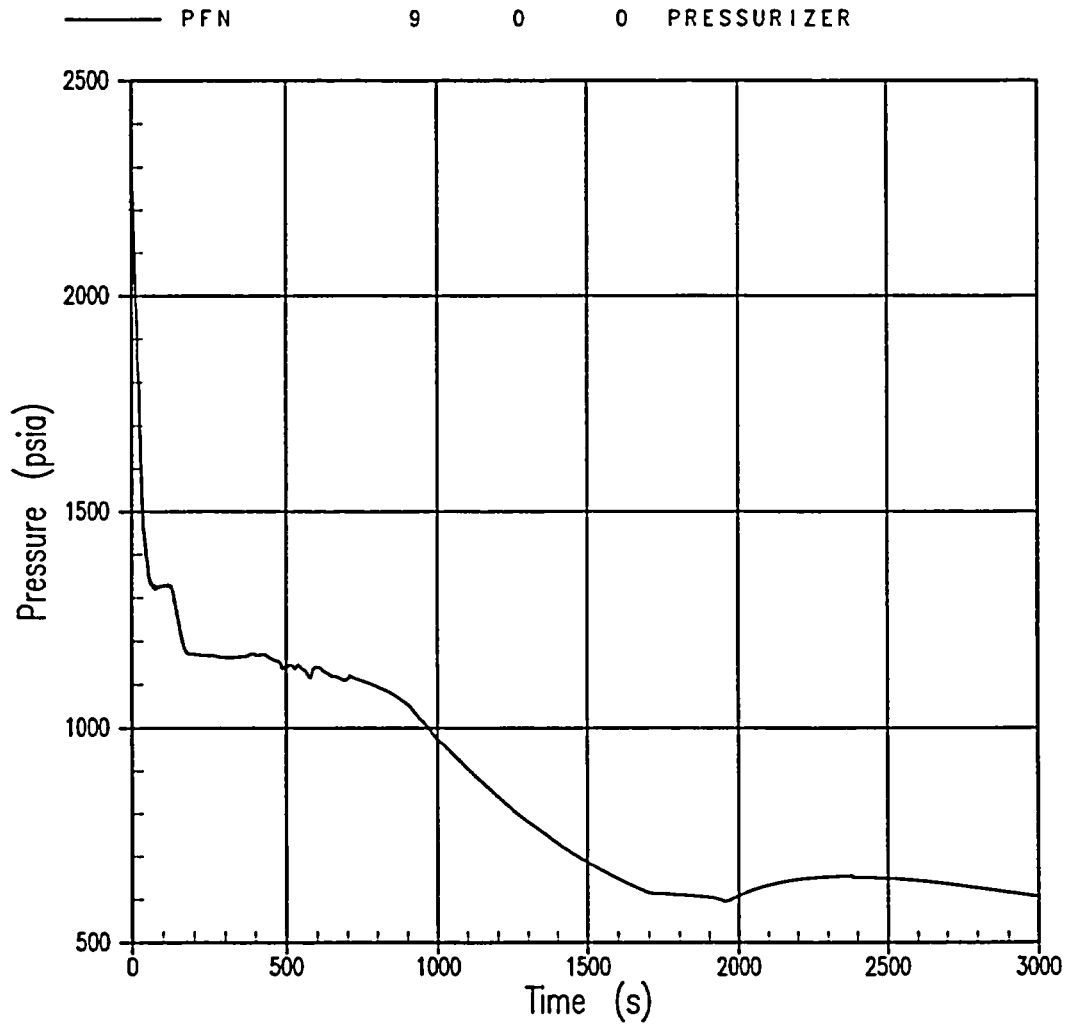


Figure 6.2-1
3-Inch Break Case, Pressurizer Pressure

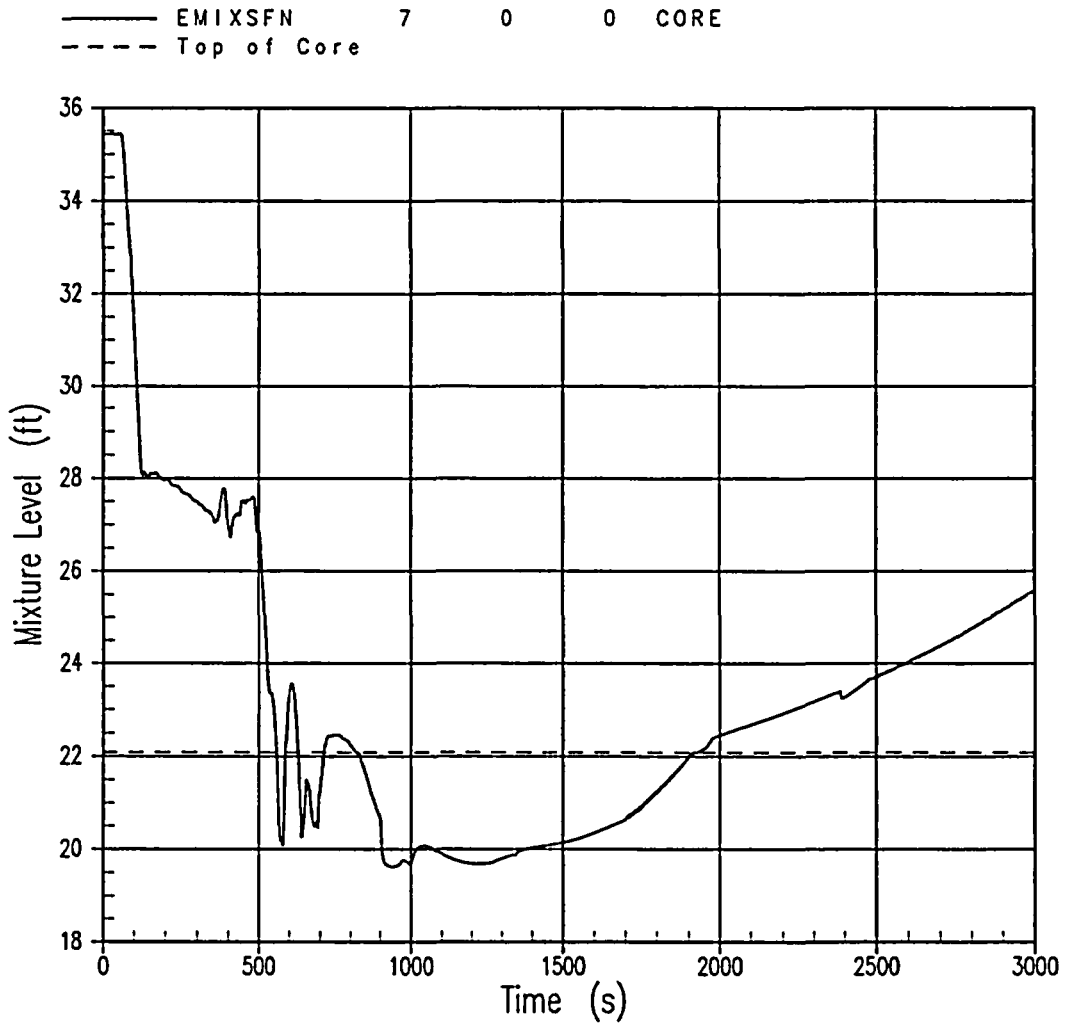


Figure 6.2-2
3-Inch Break Case, Core Mixture Level

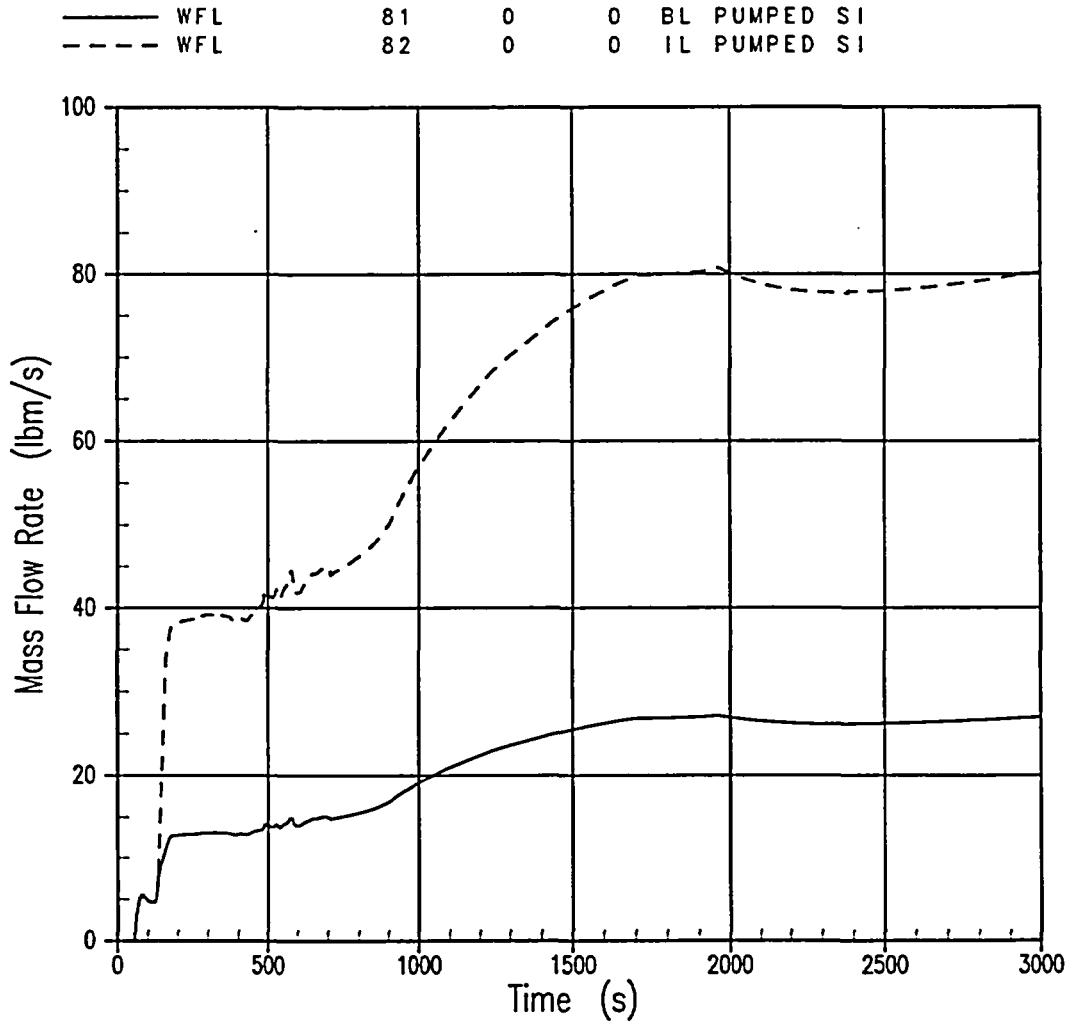


Figure 6.2-3
3-Inch Break Case, Broken Loop, and Intact Loop Pumped SI Flow Rate

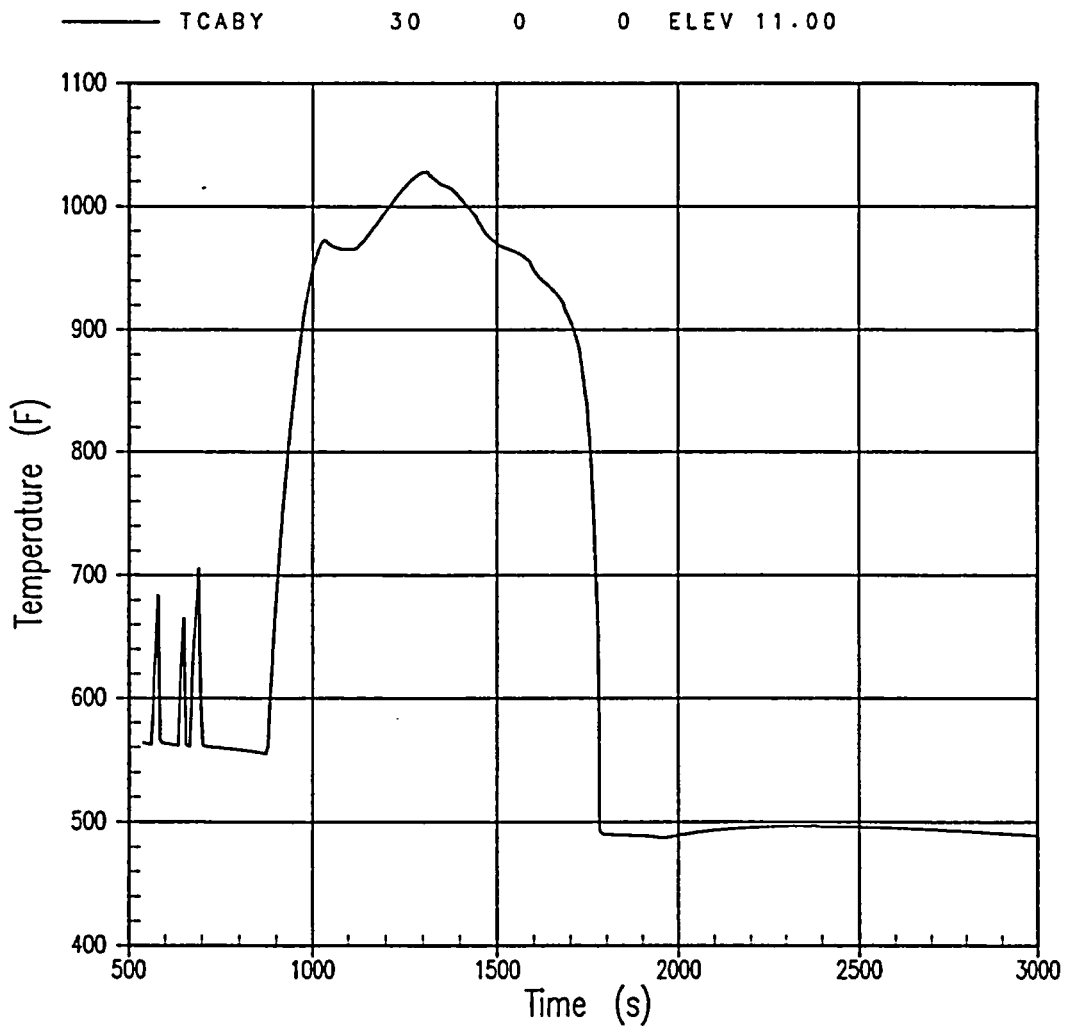


Figure 6.2-4
3-Inch Break Case, PCT at PCT Elevation (11.0 ft)

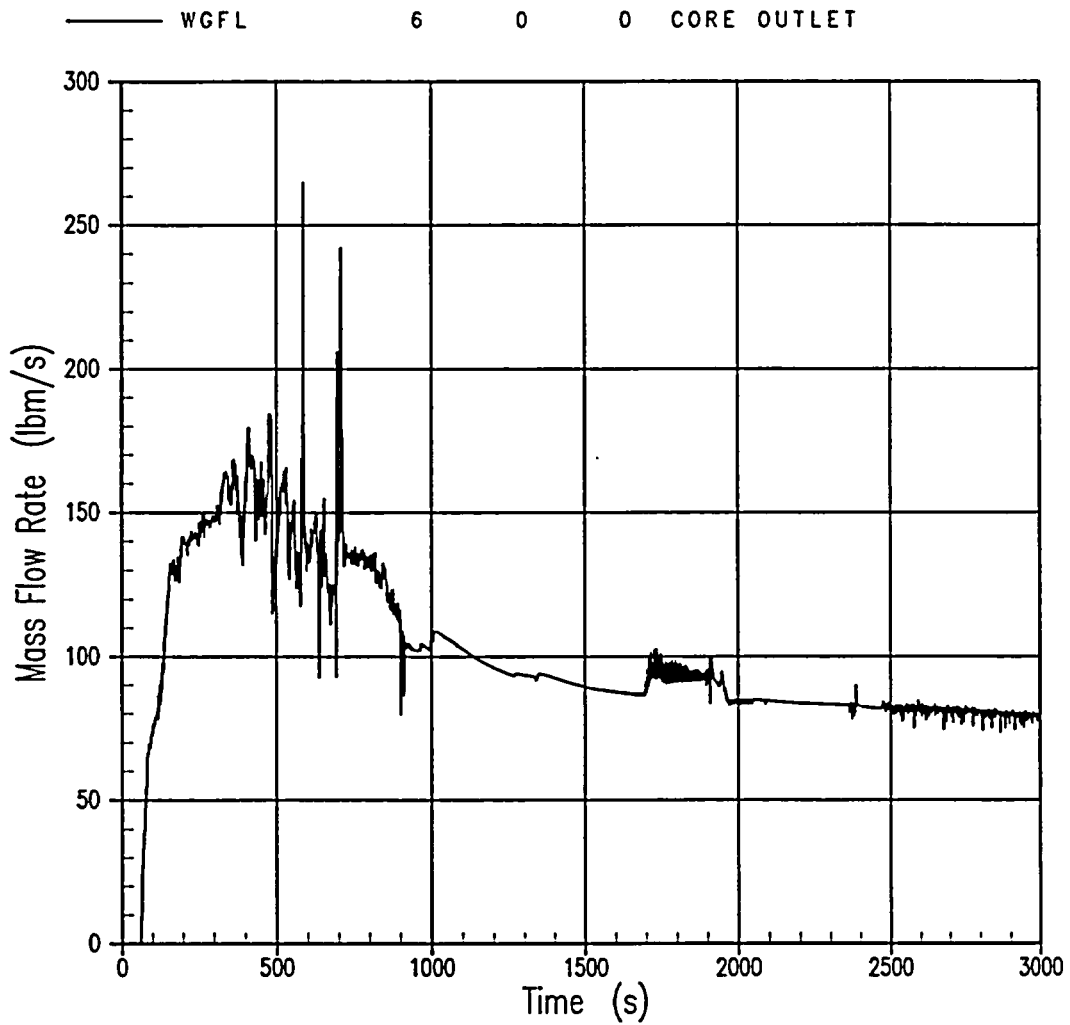


Figure 6.2-5
3-Inch Break Case, Core Exit Steam Flow

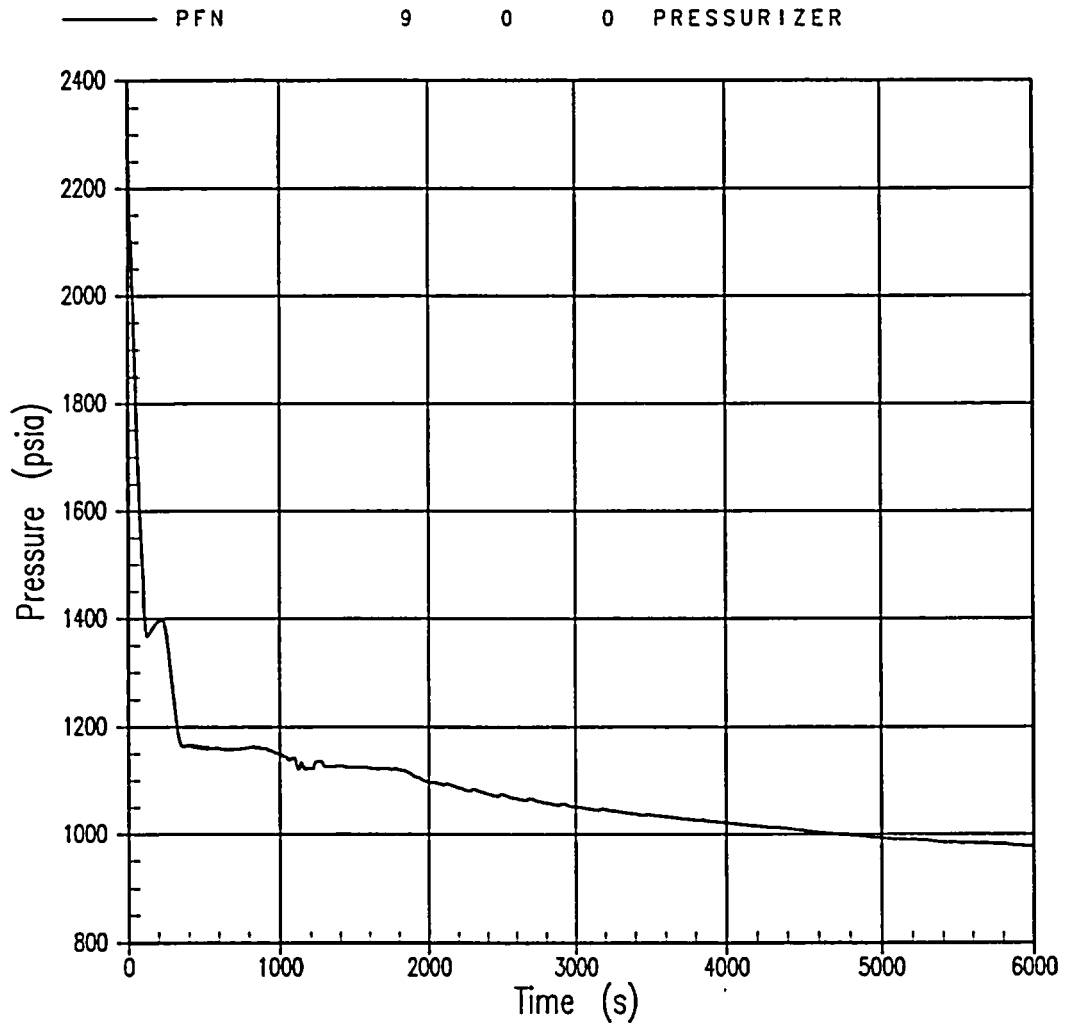


Figure 6.2-6
2-Inch Break, Pressurizer Pressure

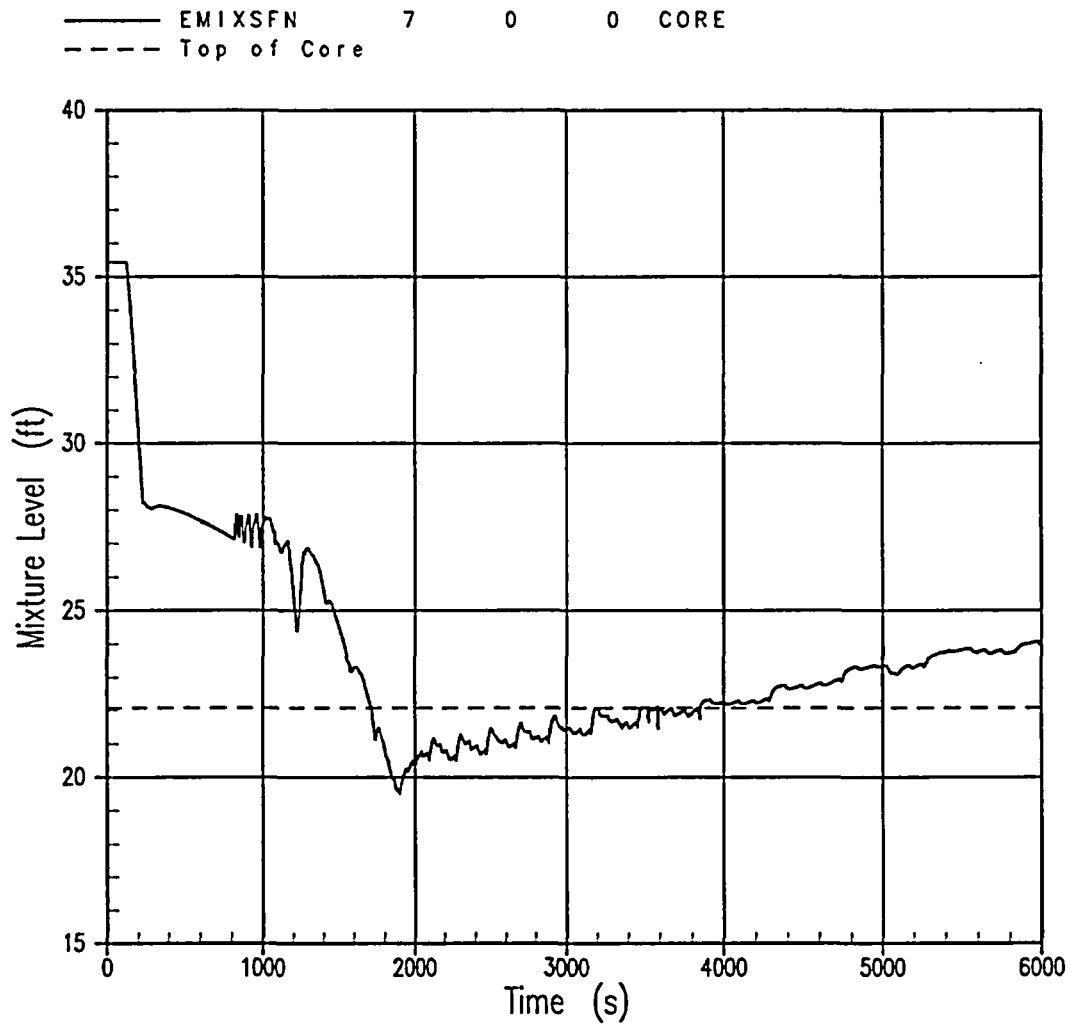


Figure 6.2-7
2-Inch Break, Core Mixture Level

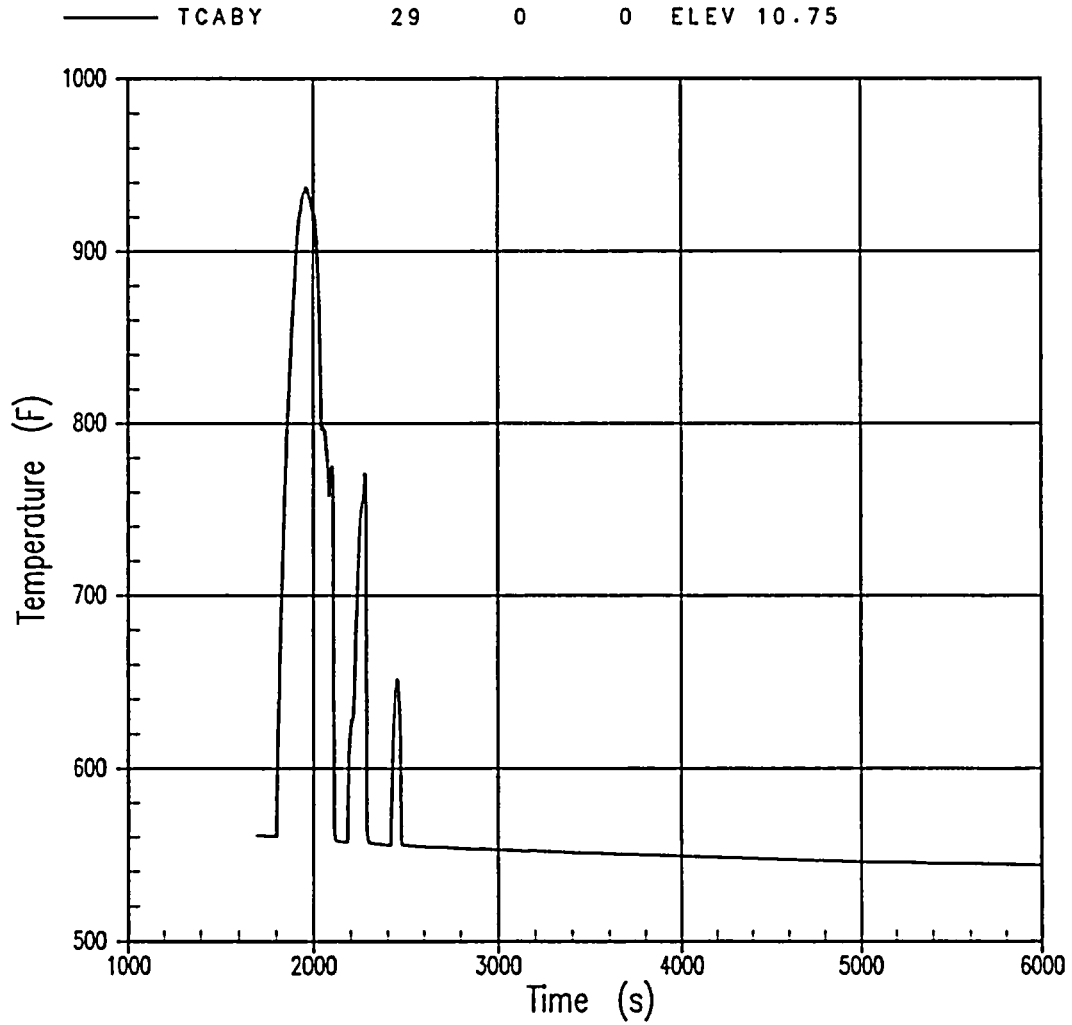


Figure 6.2-8
2-Inch Break, PCT at PCT Elevation (10.75 ft)

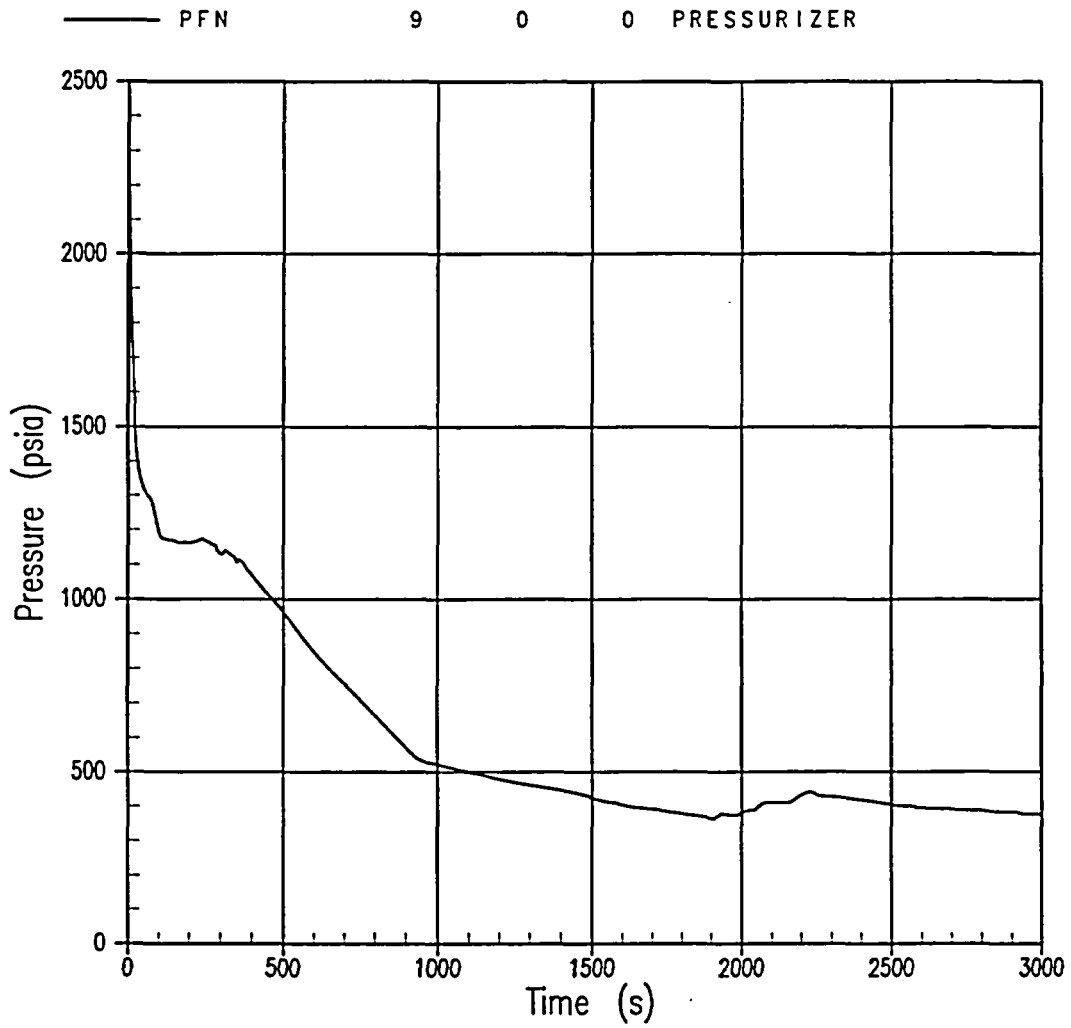


Figure 6.2-9
4-Inch Break, Pressurizer Pressure

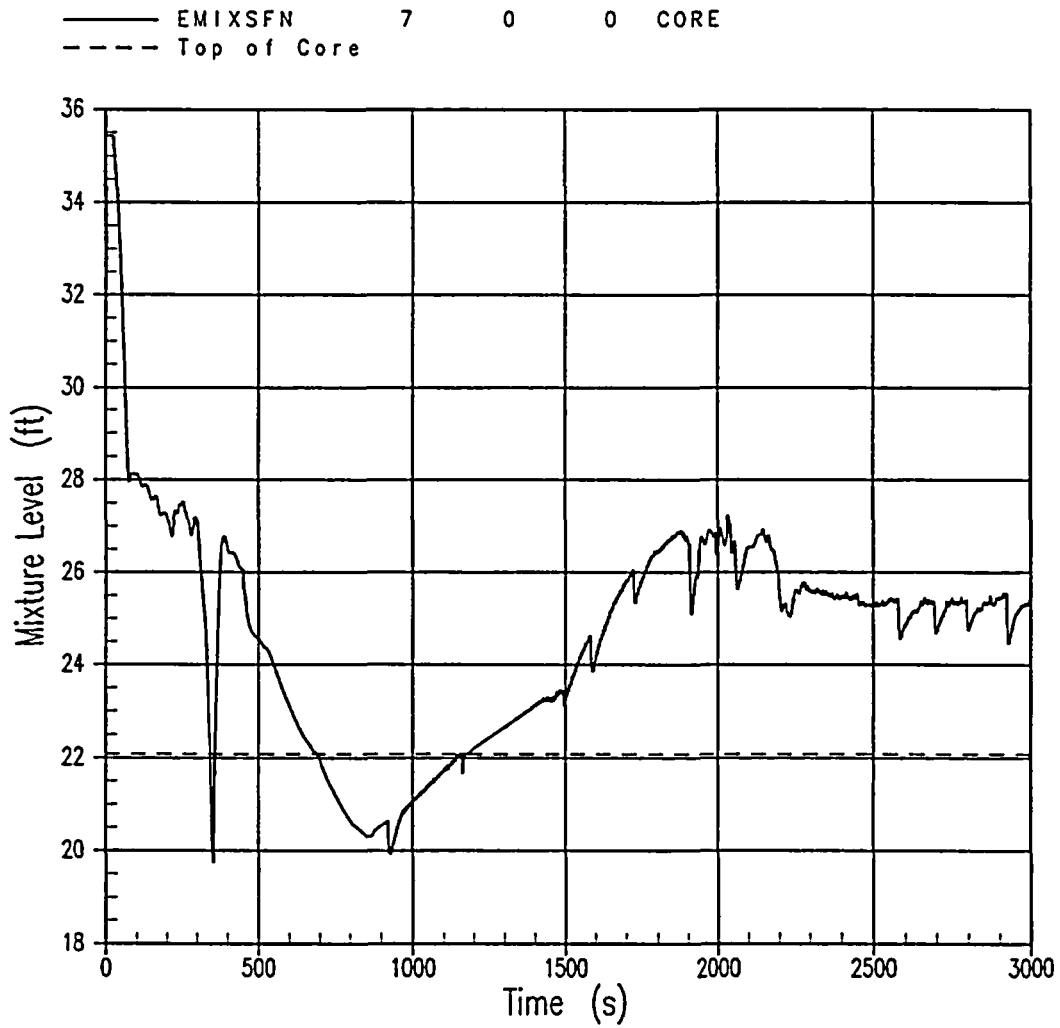


Figure 6.2-10
4-Inch Break, Core Mixture Level

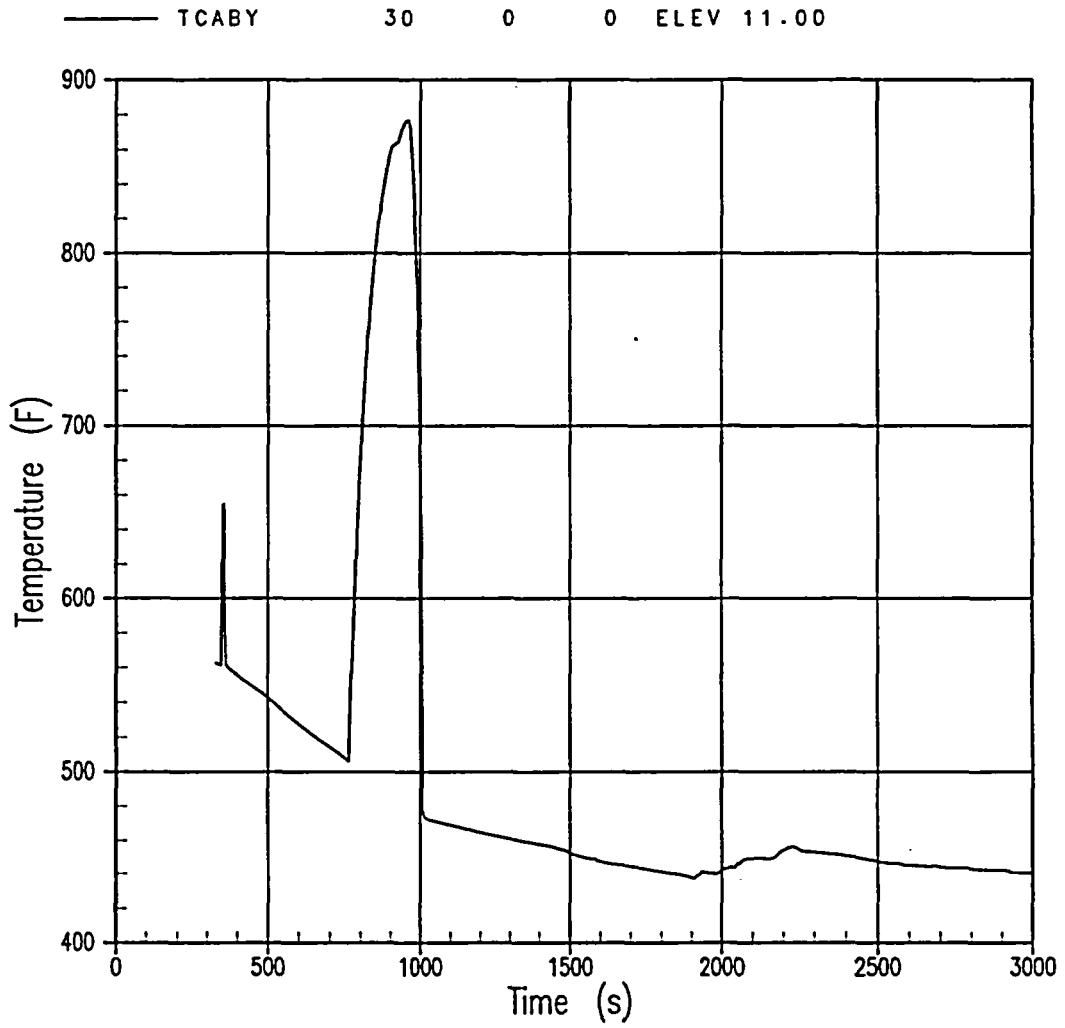


Figure 6.2-11
4-Inch Break, PCT at PCT Elevation (11.0 ft)

$$\text{Post-LOCA Sump Boron Conc. (ppm)} = 0.24 \times [\text{RCS Conc. (ppm)}] + 1808$$

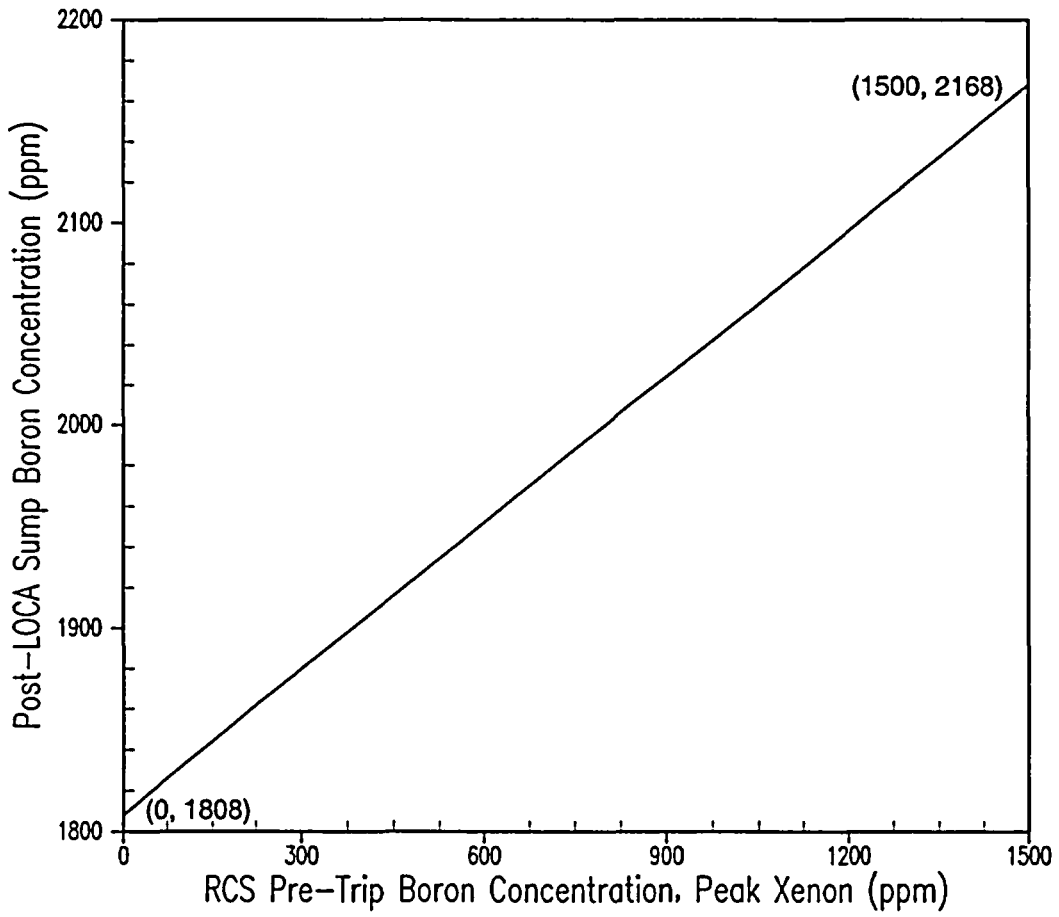


Figure 6.2-12

IP2 Uprated Post-LOCA Sump Boron Concentration Curve

6.3 Non-Loss-of-Coolant Accident Transients

6.3.1 Introduction

To support the Indian Point Unit 2 (IP2) Stretch Power Uprate (SPU) Program, all of the *Updated Final Safety Analysis Report (UFSAR)* Chapter 14 non-LOCA analyses were evaluated to determine the acceptability of plant operation at the uprated conditions. The uprated conditions addressed are those defined in Table 2.1-2 of this report for the IP2 SPU Program. The non-loss-of-coolant accident (non-LOCA) events considered herein are listed in Table 6.3-1, along with the corresponding section number in this report and the applicable UFSAR section(s).

The non-LOCA safety analysis methodology used to support the SPU was the same as that applied for the current licensing basis non-LOCA analyses. For some non-LOCA events, the SPU analyses were performed using the RETRAN-02 (RETRAN) computer code that is based on the same methods and methodology used in the current non-LOCA safety analyses using the LOFTRAN code. For certain applications, RETRAN was used in combination with other computer codes, such as VIPRE-01 for reactor core subchannel thermal-hydraulic calculations, a neutronic code such as ANC, and a fuel performance code such as PAD (as described in Section 1). RETRAN is approved for use in non-LOCA safety analysis by the NRC in the *Safety Evaluation Report (SER)* for WCAP-14882-P-A (Reference 1).

Table 6.3-1 contains a list of non-LOCA events along with the corresponding non-LOCA computer codes used. The RETRAN code has been explicitly approved by the NRC for use on each of the non-LOCA events that were analyzed using RETRAN for the SPU Program (as shown in Table 6.3-1 of this report and documented in Table 1 of the SER of WCAP-14882-P-A [Reference 1]). The RETRAN model used in the IP2 non-LOCA SPU safety analyses simulates a Westinghouse four-loop plant design, applicable to IP2, as described and presented in WCAP-14882-P-A. For each non-LOCA event analyzed, a conservative set of initial conditions and input assumptions was used to generate a conservative, plant-specific transient condition. The event and analysis conditions are provided for each non-LOCA event in subsections 6.3.2 through 6.3.15 of this document. In performing the required analyses for reload cores, Westinghouse used approved methodology (Reference 2), which provided for using conservative code input so as to bound the expected conditions for subsequent reloads.

Where applicable, the Revised Thermal Design Procedure (RTDP) methodology discussed in WCAP-11397-P-A (Reference 3) was used in the non-LOCA analyses. The RTDP methodology statistically convoluted the uncertainties of the plant operating parameters (for example, power, temperature, pressure, and flow) into the design limit departure from nucleate boiling ratio (DNBR) value. These design limit DNBR values were then used to determine the safety

analysis limit DNBR values that were used as an acceptance criterion in the DNBR-related non-LOCA analyses.

In conjunction with the SPU, the non-LOCA safety analyses supported several other related changes that directly affect the UFSAR Chapter 14 non-LOCA safety analyses. These changes are summarized in the sections that follow.

Power Upgrading

The changes in plant conditions that were considered to be directly associated with the SPU are shown in Tables 2.1-1 and 2.1-2 of this document, and discussed below.

NSSS power was increased from 3127 to 3230 MWt. This resulted in an increase in reactor power from 3115 to 3216 MWt, and a corresponding increase in rod average linear power to 6.644 kW/ft.

Thermal design flow (TDF) was maintained at 322,800 gpm for the SPU Program. The minimum measured flow (MMF), used in conjunction with the statistical RTDP departure from nucleate boiling (DNBR) methodology described in subsection 6.1.4.1, was increased to 348,300 gpm allowing for additional flow uncertainty. Core bypass flow fractions of 6.5 percent (non-statistical) and 5.9 percent (statistical) were assumed. These core bypass flow conditions were consistent with those currently supporting thimble plug elimination and, as such, would not be a change.

The maximum reactor vessel average coolant temperature (T_{avg}) was decreased from 579.2 to 572.0°F. The minimum full-power T_{avg} was also decreased from 549.4 to 549.0°F.

The non-LOCA safety analyses now support a range of main feedwater temperatures. The full-power feedwater temperature range is 390.0 to 436.2°F. The previous analyses supported a full-power feedwater temperature of 431.8°F. The feedwater temperature at hot zero power (HZP) conditions remained at 100°F. Feedwater temperatures at part-power conditions increased proportionally with power between HZP and full-power conditions.

The maximum steam generator tube plugging (SGTP) levels were decreased from 25-percent uniform/30-percent peak to 10-percent uniform for the Model 44F steam generators. Symmetric reactor coolant loop (RCL) flow conditions consistent with a maximum 10-percent uniform SGTP were assumed.

Overtemperature ΔT and Overpower ΔT Reactor Trip Setpoints

The overtemperature ΔT and overpower ΔT (OT ΔT /OP ΔT) reactor trip functions were assumed to be available in several non-LOCA transient analyses to ensure that the departure from nucleate boiling (DNB) design basis and the fuel centerline melting design basis would be satisfied. The OT ΔT and OP ΔT reactor trip safety analysis setpoints were generated assuming steady-state conditions and were based on a number of inputs, including the nominal core thermal power and the core thermal safety limits. The core thermal safety limits were the locus of core inlet temperature conditions for a range of powers and a range of pressures, for which the DNBR was equal to the limit value.

As a result of the increased core thermal power for the SPU, the safety analysis limit DNBR and core thermal safety limits were revised, resulting in a change to the OT ΔT and OP ΔT reactor protection trip setpoints. The safety analysis limit DNBR was revised from 1.58 (typical and thimble cell) to 1.48 (typical and thimble cell). The revised core thermal safety limits presented in Figure 6.3-1 (and Figure 1 of the Core Operating Limits Report [COLR]) were based on the SPU conditions defined in Table 2.1-2 of this report.

The safety analysis values for the OT ΔT and OP ΔT reactor protection trip setpoints, based on the revised core thermal safety limits, were as follows:

Overtemperature ΔT Reactor Trip Setpoint

$$\Delta T \leq \Delta T_0 [K_1 - K_2 [(1 + \tau_1 s) / (1 + \tau_2 s)] (T_{avg} - T') + K_3 (P - P') - f(\Delta I)]$$

Where: $K_1 = 1.42$
 $K_2 = 0.020/^\circ\text{F}$
 $K_3 = 0.00070/\text{psi}$
 $\tau_1 = 25.0$ seconds
 $\tau_2 = 3.0$ seconds
 $T' \leq 572^\circ\text{F}$
 $P' = 2235$ psig

$K_1 =$ Preset manually adjustable bias (fraction)

$K_2 =$ Preset manually adjustable gain based on the effect of temperature on the design limits ($1/^\circ\text{F}$)

K_3 = Preset manually adjustable gain based on the effect of pressure on the design limits (1/psi)

ΔT_o = Reference ΔT , measured at nominal full power for the channel being calibrated ($^{\circ}F$)

T_{avg} = Measured average temperature for each calibrated channel (input from instrument racks) ($^{\circ}F$)

T' = Reference T_{avg} , measured at nominal full power for the channel being calibrated ($^{\circ}F$)

P = Measured pressurizer pressure (input from instrument racks) (psig)

P' = Pressurizer pressure at nominal full power (psig)

$f(\Delta I)$ = Function of the indicated difference between the top and bottom detectors of the power range nuclear ion detectors (see below)

$(1 + \tau_1 s)/(1 + \tau_2 s)$ = Lead/lag compensation

Where: τ_1 = Preset manually adjustable dynamic compensation time constant (Lead for OT ΔT trip setpoint) (seconds)

τ_2 = Preset manually adjustable dynamic compensation time constant (Lag for OT ΔT trip setpoint) (seconds)

s = Laplace transform operator (seconds $^{-1}$)

$f(\Delta I)$ Function for Overtemperature ΔT

Function of the indicated difference between the top and bottom detectors of the power range nuclear ion detectors:

- For each percent that ΔI is < -30.0 percent, reduce the OT ΔT trip setpoint by the equivalent of 1.97 percent RTP rated thermal power (RTP).
- For ΔI between -30.0 percent and $+7.0$ percent, the OT ΔT $f(\Delta I)$ function is equal to 0.0.
- For each percent that ΔI is $> +7.0$ percent, reduce the OT ΔT trip setpoint by the equivalent of 2.25 percent RTP.

Overpower ΔT Reactor Trip Setpoint

$$\Delta T \leq \Delta T_o [K_4 - K_5 [(\tau_3 s) / (1 + \tau_3 s)] (T_{avg}) - K_6 (T_{avg} - T'')]$$

Where: $K_4 = 1.164$

$K_5 = 0.0/^\circ\text{F}$ for decreasing T_{avg} ; and
 $= 0.0188/^\circ\text{F}$ for increasing T_{avg}

$K_6 = 0.0/^\circ\text{F}$ for $T_{avg} \leq T''$; and
 $= 0.0015/^\circ\text{F}$ for $T_{avg} > T''$

$\tau_3 = 10.0$ seconds

$T'' \leq 572^\circ\text{F}$

$K_4 =$ Preset manually adjustable bias (fraction)

$K_5 =$ Preset manually adjustable gain that compensates for piping and thermal time delays ($1/^\circ\text{F}$)

$K_6 =$ Preset manually adjustable gain that accounts for the effects of coolant density and heat capacity on the relationship between ΔT and thermal power ($1/^\circ\text{F}$)

$\Delta T_o =$ Reference ΔT , measured at nominal full power for the channel being calibrated ($^\circ\text{F}$)

$T_{avg} =$ Measured average temperature for each calibrated channel (input from instrument racks) ($^\circ\text{F}$)

$T'' =$ Reference T_{avg} , measured at nominal full power for the channel being calibrated ($^\circ\text{F}$)

$(\tau_3 s)/(1 + \tau_3 s) =$ Rate/lag compensation

Where: $\tau_3 =$ Preset manually adjustable dynamic compensation time constant (rate lag time constant for OP ΔT trip setpoint) (seconds)

$s =$ Laplace transform operator (seconds⁻¹)

The safety analysis values assumed for the time constants (first order lags) on the measurements of T_{avg} and ΔT used in the OT ΔT and OP ΔT equations are 4.0 seconds.

The nominal values assumed for T_{avg} and pressure in the OTΔT and OPΔT setpoint calculations bound the SPU Program conditions for a nominal operating T_{avg} from 549.0 to 572.0°F.

With respect to Reactor Coolant System (RCS) pressure, the OTΔT and OPΔT reactor trip functions were applicable for a range of pressurizer pressures from 1860 to 2440 psia. This analyzed range bounds pressure conditions between the low- and high-pressurizer pressure reactor trip settings with consideration given to the appropriate uncertainty.

To ensure proper operation of the OTΔT and OPΔT reactor trip functions over the range of applicable RCS temperatures, the instrumentation must be capable of measuring temperatures over the following ranges:

$$\begin{aligned} 510^{\circ}\text{F} &\leq T_{\text{cold}} \leq 593^{\circ}\text{F} \\ 548^{\circ}\text{F} &\leq T_{\text{avg}} \leq 615^{\circ}\text{F} \\ 585^{\circ}\text{F} &\leq T_{\text{hot}} \leq 637^{\circ}\text{F} \end{aligned}$$

The effect of the change in the core thermal safety limits as well as resulting change in the OTΔT and OPΔT reactor protection trip setpoints are addressed for non-LOCA transients in the evaluations and analyses described in the following sections.

Auxiliary Feedwater

To support the SPU Program, a requirement was added for delivery of additional auxiliary feedwater (AFW) to preclude a pressurizer water-solid condition for the loss of normal feedwater (LONF) and loss of all AC power (LOAC) to the station auxiliaries event analyses. The LONF and LOAC events address a LONF (from pump failures, valve malfunctions, or LOAC), which resulted in a reduction in capability of the secondary system to remove heat generated in the reactor core. If an alternative supply of feedwater is not supplied to the plant, residual heat following a reactor trip may heat the primary system water to the point at which water relief from the pressurizer could occur, potentially generating a more serious plant condition without other incidents occurring independently. To ensure acceptable results were obtained in the LONF/LOAC event analyses (addressed in subsections 6.3.7 and 6.3.8), operator action was assumed at 10 minutes following reactor trip to align an additional train of AFW (aside from the single motor-driven AFW train automatically actuated on a low-low steam generator water level signal).

Neutronics/Reactivity Modeling

To support future reload design activities with uprated core power and to support several event-specific uprating analyses, several neutronics-related analysis input assumptions were changed to support the SPU.

To provide margin for future reload design activities, the change in boron concentration from maximum critical boron concentration (all rods inserted) to a critical boron concentration, a k-effective of < 0.95 for the Mode 6 (refueling) boron dilution analysis was increased from 610 to 660 ppm. The Mode 6 boron dilution analysis is presented in subsection 6.3.5 of this document.

To facilitate future reload evaluations, the isothermal temperature coefficient, ejected rod worth, delayed neutron fraction, and Doppler defect values assumed in the rod ejection analysis were adjusted to offset the effect the power uprate has on the end-of-life/hot zero power (EOL/HZP) Case. The ejected rod worth was reduced from 800 to 790 pcm and the delayed neutron fraction was relaxed from 0.0040 to 0.0042. The rod ejection analysis is presented later in subsection 6.3.15.

To support the uncontrolled rod cluster control assembly (RCCA) withdrawal at power analysis, the maximum reactivity insertion rate was limited to ≤ 70 pcm/sec (93.31 pcm/in), corresponding to maximum differential RCCA worth at maximum RCCA withdrawal rate. This is discussed in subsection 6.3.3.

Pressurizer Safety Valve Setpoint Tolerance Increase

The pressurizer code safety valve setpoint tolerance applicable to the safety analysis was relaxed to account for drift during the fuel cycle. The lift setting for the pressurizer safety valves (PSVs) was calibrated to ± 1 percent, however the non-LOCA safety analyses for the uprating are performed to support an "as-found" tolerance of +3 percent / -2 percent.

Fuel Temperatures

Revised fuel temperatures generated in support of the SPU conditions were applied as appropriate in the non-LOCA safety analyses.

Reactor Trip

There were various instrumentation delays associated with each reactor trip function that was modeled directly in the non-LOCA safety analyses. The total delay time was defined as the time from when trip conditions are reached to the time the rods are free to fall. The safety analysis

trip setpoint and maximum time delay assumed in the non-LOCA safety analysis for each reactor trip function at SPU conditions are shown in Table 6.3-2.

Table 6.3-3 summarizes key analysis assumptions considered in the IP2 SPU non-LOCA analyses and evaluations.

Event Classification

The non-LOCA accidents are classified by the American Nuclear Society (ANS) as Condition II, III, or IV events. The ANS categorizes events based upon expected frequency of occurrence and severity as follows.

- Condition I: Normal operation and operational transients
- Condition II: Faults of moderate frequency
- Condition III: Infrequent faults
- Condition IV: Limiting faults

Condition I events are normal operation incidents that are expected to occur frequently or regularly. These occurrences are accommodated with margin between any plant parameter and the value of that parameter that would require either automatic or manual protective action.

Condition II events (which are the majority of the non-LOCA events) are incidents of moderate frequency that may reasonably occur during a calendar year of operation. These faults, at worst, result in a reactor trip with the plant capable of returning to power operations after corrective actions. Condition II incidents will not generate a more serious accident (Condition III or IV) without other incidents occurring independently.

Condition III events are infrequent faults that may reasonably occur during the lifetime of a plant. These faults will not cause more than a small fraction of fuel elements to be damaged. No consequential loss of function of the RCS or containment as fission product barriers can occur. The release of radioactive materials to unrestricted areas may exceed 10CFR20 limits; however, they will not be enough to interrupt or restrict public use of those areas beyond the exclusion radius. Condition III incidents will not generate a more serious accident (Condition IV) without other incidents occurring independently.

Condition IV events are limiting faults that are not expected to occur but are postulated because their consequences would include the potential for significant radioactive releases. The release of radioactive material will not result in an undue risk to public health and safety exceeding the guidelines of 10CFR100. No consequential loss of function of systems required to mitigate the event can occur.

The results of all analyses and evaluations demonstrated that applicable safety analysis acceptance criteria were satisfied at the SPU conditions detailed in Table 2.1-2 of this report.

6.3.2 Uncontrolled RCCA Withdrawal from a Subcritical or Low Power Startup Condition

6.3.2.1 Introduction

An RCCA withdrawal incident is defined as an uncontrolled addition of reactivity to the reactor core by withdrawal of control rods resulting in a power excursion. While the probability of a transient of this type is extremely low, operator action or a malfunction of the Reactor Control Rod Drive System could cause such a transient. This could occur with the reactor either subcritical or at power. The at-power case is discussed later in subsection 6.3.3.

Reactivity is added at a prescribed and controlled rate in bringing the reactor from shutdown to low power during startup by RCCA withdrawal or by reducing the reactor coolant boron concentration. RCCA motion can cause much faster changes in reactivity than could occur from changing the boron concentration.

The RCCA drive mechanisms are wired into pre-selected bank configurations that remain the same throughout reactor life. These circuits prevent the RCCAs from being automatically withdrawn in other than their respective banks. Power supplied to the banks is controlled so that no more than 2 banks can be withdrawn at the same time and in their defined withdrawal sequence. The RCCA drive mechanisms are the magnetic latch type, and coil actuation is sequenced to provide variable speed rod travel. The maximum reactivity insertion rate is analyzed in the detailed plant analysis assuming simultaneous withdrawal of the combination of the 2 sequential control banks having the maximum combined worth at maximum speed. The maximum reactivity insertion rate, even with these assumptions, is well within the capability of the Reactor Protection System (RPS) to prevent core damage.

Should a continuous RCCA withdrawal be initiated, the following automatic features of the RPS will terminate the transient:

- Source range neutron flux reactor trip - actuated when either of 2 independent source range channels indicate above a pre-selected, manually adjustable setpoint. This trip function can be manually bypassed only after either of 2 intermediate range flux channels indicate above a specified level. It is automatically reinstated when both intermediate channels indicate below a specified level.

- Intermediate-range neutron-flux reactor trip - actuated when either of 2 independent intermediate range channels indicate above a pre-selected, manually adjustable setpoint. This trip function can be manually bypassed only after 2 of 4 power range channels indicate above approximately 10 percent full-power. It is automatically reinstated when 3 of 4 channels indicate below this value.
- Power-range high-neutron-flux reactor trip (low setting) - actuated when 2 of 4 power range channels indicate above approximately 25 percent full-power. This trip function can be manually bypassed when 2 of 4 power range channels indicate above approximately 10 percent full-power. It is automatically reinstated when 3 of 4 channels indicate below this value.
- Power-range high-neutron-flux reactor trip (high setting) - actuated when 2 of 4 power range channels indicate above a preset setpoint. This trip function is always active.

The neutron flux response to a continuous reactivity insertion is characterized by a very fast initial increase terminated by the reactivity feedback effect of the negative Doppler power coefficient. This self-limitation of the initial power increase is of prime importance since it limits nuclear power to an acceptable level prior to protection system action. After the initial increase, the nuclear power is momentarily reduced and then, if the incident is not terminated by a reactor trip, the nuclear power increases again, but at a much slower rate.

Termination of the startup transient by the above protection channels prevents core damage. In addition, control rod stops on high intermediate range flux level (1 of 2) and high power range flux level (1 of 4) serve to halt rod withdrawal and prevent the need to actuate the intermediate range flux level trip and power range flux level trip, respectively.

Note: Automatic rod withdrawal has been physically disabled at IP2.

6.3.2.2 Input Parameters and Assumptions

The Standard Thermal Design Procedure (STDP) was used in the accident analysis. To obtain conservative results for the analysis, the following assumptions were made concerning initial reactor conditions:

- Since the magnitude of the nuclear power peak reached during the initial part of the transient, for any given rate of reactivity insertion, is strongly dependent on the Doppler power reactivity coefficient, a conservatively low (least negative) value was used.

- The contribution of the moderator reactivity coefficient is negligible during the initial part of the transient because heat transfer time between the fuel and moderator is much longer than nuclear flux response time. However, after the initial neutron flux peak, the succeeding rate of power increase is affected by the moderator reactivity coefficient. Accordingly, the most positive moderator temperature coefficient was assumed since this yields the maximum rate of power increase.
- The analysis assumed the reactor to be at HZP conditions with a nominal temperature of 547°F. This assumption is more conservative than that of a lower initial system temperature (that is, shutdown conditions) because it yields a larger fuel-to-moderator heat transfer coefficient, a larger specific heat of both moderator and fuel, and a less-negative (smaller absolute magnitude) Doppler coefficient. The less-negative Doppler coefficient reduces the Doppler feedback effect, thereby increasing the neutron flux peak. The high neutron flux peak combined with a high fuel-specific heat and larger heat transfer coefficient yields a larger peak heat flux. The analysis also assumes the initial effective multiplication factor (K_{eff}) to be 1.0 since this results in the maximum neutron flux peak.
- Reactor trip is assumed on power-range high-neutron flux (low setting). The most adverse combination of instrumentation error, setpoint error, delay for trip signal actuation, and delay for control rod assembly release is taken into account. The analysis assumes a 10-percent uncertainty in power range flux trip setpoint (low setting), raising it from the nominal value of 25 to 35 percent. During the transient, the rise in nuclear power is so rapid that the effect of error in the trip setpoint on the actual time of rod release is negligible. In addition, total reactor trip reactivity is based on the assumption that the highest worth control rod assembly is stuck in its fully withdrawn position.
- The maximum positive reactivity insertion rate assumed is greater than that for simultaneous withdrawal of the 2 sequential control banks having the greatest combined worth at maximum speed (45 inch/min, which corresponds to 72 steps/min).
- The DNB analysis assumes the most limiting axial and radial power shapes associated with having the 2 highest combined worth banks in their high-worth position.
- The analysis assumes initial power to be below that expected for any shutdown condition (10^{-9} fraction of nominal power). The combination of highest reactivity insertion rate and low initial power produces the highest peak heat flux.

- The analysis assumes 2 reactor coolant pumps (RCPs) in operation. This is conservative with respect to the DNB transient.

6.3.2.3 Description of Analysis

The analysis of the uncontrolled-RCCA-bank-withdrawal-from-subcriticality was performed in 3 stages. First, a spatial neutron kinetics computer code, TWINKLE (Reference 4), was used to calculate the core average nuclear power transient, including various core feedback effects, that is, Doppler and moderator reactivity. Next, the FACTRAN computer code (Reference 5) used the average nuclear power calculated by TWINKLE and performed a fuel rod transient heat transfer calculation to determine average heat flux and temperature transients. Finally, the average heat flux calculated by FACTRAN was used in the VIPRE-W computer code for DNBR calculations.

6.3.2.4 Acceptance Criteria

The uncontrolled-RCCA-bank-withdrawal-from-subcritical event was considered an ANS Condition II event, a fault of moderate frequency, and was analyzed to ensure that the core and RCS were not adversely affected. This was demonstrated by showing that the minimum DNBR remained above the applicable safety analysis limit and that peak fuel centerline temperature remained within acceptable limits.

6.3.2.5 Results

The results of the uncontrolled-RCCA-bank-withdrawal analysis performed at the SPU conditions showed that the minimum DNBR remained above the safety analysis limit at all times (see discussion in subsection 7.2.3.2.6) and that peak fuel centerline temperature remained below that at which fuel melt occurs. The calculated sequence of events is shown in Table 6.3-4. The neutron flux transient, thermal flux transient, and the peak fuel centerline and clad temperature transients for this accident are shown in Figures 6.3-2 through 6.3-5, respectively.

6.3.2.6 Conclusions

In the event of an RCCA withdrawal incident from the subcritical condition, the core and RCS would not be adversely affected since the combination of thermal power and coolant temperature resulted in a minimum DNBR greater than the safety analysis limit. Furthermore, since the maximum fuel temperatures predicted to occur during this event were much less than those required for fuel melting (4800°F), no fuel damage was predicted as a result of this transient at SPU conditions. Clad damage was also precluded.

6.3.3 Uncontrolled RCCA Assembly Withdrawal at Power

6.3.3.1 Introduction

An uncontrolled-RCCA-bank-withdrawal-at-power event that causes an increase in core heat flux could be the result of an operator error or a malfunction in the Rod Control System. Immediately following initiation of the accident, the steam generator heat removal rate lags behind the core power generation rate. This imbalance between heat removal and heat generation rate causes the reactor coolant temperature to rise. Unless terminated, the power increase and resultant coolant temperature rise could eventually result in DNB and/or fuel centerline melt. Therefore, to avoid damage to the core, the RPS is designed to automatically terminate the transient before the DNBR falls below the safety analysis limit or the fuel rod linear heat generation rate (kW/ft) limit is exceeded.

The automatic RPS features that prevent core damage in an RCCA-bank-withdrawal-incident at-power by actuating a reactor trip include the following:

- Any 2-out-of-4 power range high neutron flux channels exceed an overpower setpoint.
- Any 2-out-of-4 ΔT channels exceed an OT ΔT setpoint. This setpoint is automatically varied with axial power distribution, coolant average temperature, and coolant average pressure to protect against DNB.
- Any 2-out-of-4 ΔT channels exceed an OP ΔT setpoint. This setpoint is automatically varied with coolant average temperature so that the allowable heat generation rate (kW/ft) is not exceeded.
- Any 2-out-of-4 high-pressurizer pressure channels exceed a fixed setpoint. This setpoint is less than the set pressure for the PSVs.
- Any 2-out-of-3 high-pressurizer water level channels exceed a fixed setpoint.

In addition to the above listed reactor trips, there are several RCCA bank withdrawal blocks that are not credited in the accident analyses but would serve to limit the severity of this event. These are:

- High neutron flux (1-out-of-4 power range channels)
- OT ΔT (1-out-of-4 channels)
- OP ΔT (1-out-of-4 channels)

6.3.3.2 Input Parameters and Assumptions

A number of cases were analyzed assuming a range of reactivity insertion rates for both minimum and maximum reactivity feedback conditions at various power levels. The cases presented in subsection 6.3.3.5 are representative for this event.

For an uncontrolled-RCCA-bank-withdrawal-at-power accident, the following conservative assumptions are made:

- This accident is analyzed with the RTDP (Reference 3). Therefore, initial reactor power, pressurizer pressure, and RCS temperatures are assumed to be at their nominal values. Uncertainties in initial conditions are included in the DNBR limit.
- For reactivity coefficients, two cases are analyzed.
 - Minimum Reactivity Feedback: A zero moderator density coefficient and a least-negative Doppler-only power coefficient form the basis for the BOL minimum reactivity feedback assumption.
 - Maximum Reactivity Feedback: A conservatively large positive moderator density coefficient of $0.5 \Delta k/g/cm^3$ (corresponding to a large negative MTC) and a most-negative Doppler-only power coefficient formed the basis for the EOL maximum reactivity feedback assumption.
- The reactor trip on high neutron flux is actuated at a value of 116-percent nominal full power. The ΔT trips included all adverse instrumentation and setpoint errors, with maximum delay for trip signal actuation.
- The RCCA trip insertion characteristic is based on the assumption that the highest worth RCCA is stuck in its fully withdrawn position.
- A range of reactivity insertion rates is examined. The maximum positive reactivity insertion rate is greater than that which would be obtained from the simultaneous withdrawal of the 2 control rod banks having the maximum combined differential rod worth at a conservative speed (45 inches/minute, which corresponds to 72 steps/minute).
- Initial power levels of 10, 60, and 100 percent was considered.

- The effect of a full-power RCS T_{avg} window was considered for the uncontrolled-RCCA-bank-withdrawal-at-power analysis. A conservative calculation modeling the high end of the T_{avg} window was explicitly analyzed since this is limiting with respect to the DNBR results.
- The effect of a feedwater temperature window was also considered. The low end of the feedwater temperature window was determined to be limiting with respect to the DNBR results.

6.3.3.3 Description of Analysis

This analysis demonstrated how the protection functions actuate for various combinations of reactivity insertion rates and initial conditions.

The rod-withdrawal-at-power event was analyzed with the RETRAN computer code (Reference 1). The program simulates the neutron kinetics, RCS, pressurizer, pressurizer relief and safety valves, pressurizer spray, steam generators, and main steam safety valves (MSSVs). The program computed pertinent plant variables including temperatures, pressures, power level, and DNBR.

6.3.3.4 Acceptance Criteria

Based on its frequency of occurrence, the uncontrolled-RCCA-bank-withdrawal-at-power accident is considered a Condition II event as defined by the ANS. The following items summarize the main acceptance criteria associated with this event.

The critical heat flux should not be exceeded. This was ensured by demonstrating that the minimum DNBR did not go below the limit value at any time during the transient.

Pressure in the RCS and Main Steam System (MSS) should be maintained below 110 percent of design pressures. With respect to peak pressure, the uncontrolled-RCCA-bank withdrawal-at-power-accident was bounded by the loss-of-external-electrical-load analysis described in subsection 6.3.6.

The protection features presented in subsection 6.3.3.1 provided mitigation of the uncontrolled-RCCA-bank-withdrawal-at-power transient so that the above criteria were satisfied.

6.3.3.5 Results

Figures 6.3-6 through 6.3-11 show the transient response for a rapid uncontrolled-RCCA-bank-withdrawal incident (70 pcm/sec) starting from 100-percent power with minimum feedback. Reactor trip on high neutron flux occurred shortly after the start of the accident. Because of the rapid change in nuclear power with respect to the thermal time constants of the fuel, an immediate reactor trip ensured margin to the minimum DNBR limit was maintained.

The transient response for a slow uncontrolled RCCA bank withdrawal (1 pcm/s) from 100-percent power with minimum feedback is shown in Figures 6.3-12 through 6.3-17. Reactor trip on OTΔT occurred after a much longer period, and the temperature rise was consequently larger, than for a rapid RCCA bank withdrawal. Again, the minimum DNBR was greater than the safety analysis limit.

Figure 6.3-18 shows the minimum DNBR as a function of reactivity insertion rate from 100-percent power for both minimum and maximum reactivity feedback conditions. The high neutron flux and OTΔT reactor trip functions provided DNB protection over the range of reactivity insertion rates. The minimum DNBR was never less than the safety analysis limit.

Figures 6.3-19 and 6.3-20 show the minimum DNBR as a function of reactivity insertion rate for RCCA-bank-withdrawal incidents starting at 60- and 10-percent power, respectively. The results were similar to the 100-percent power case. However, as the initial power level decreased, the range over which the OTΔT trip was effective increased. In no case did the DNBR fall below the safety analysis limit.

The calculated sequence of events for the two cases illustrated in the figures is shown in Table 6.3-5. With the reactor tripped, the plant eventually returned to a stable condition. The plant can subsequently be cooled down further by following normal plant shutdown procedures.

6.3.3.6 Conclusions

The high neutron flux and OTΔT reactor trip functions provided adequate protection over the entire range of possible reactivity insertion rates, that is, the minimum value of DNBR was always larger than the safety analysis limit. The RCS and MSS were maintained below 110 percent of their design pressures. Therefore, the results of the analysis showed that an uncontrolled-RCCA-withdrawal-at power did not adversely affect the core, RCS, or MSS, and all applicable acceptance criteria were met.

6.3.4 RCCA Drop/Misoperation

6.3.4.1 Introduction

RCCA misoperation accidents include the following:

- One or more dropped RCCAs within the same group
- A dropped RCCA bank
- A statically misaligned RCCA

Each RCCA has a position indicator channel that displays the position of the assembly in a display grouping that is convenient to the operator. Fully inserted RCCAs are also indicated by a rod-at-bottom signal that actuates a local alarm and control room annunciator. Group demand position is also indicated.

RCCAs move in preselected banks, and the banks always move in the same preselected sequence. Each bank of RCCAs consists of two groups. The rods comprising a group operate in parallel through multiplexing thyristors. The two groups in a bank move sequentially so that the first group is always within one step of the second group in the bank. A definite schedule of actuation (or de-actuation) of the stationary gripper, movable gripper, and lift coils of the control rod drive mechanism (CRDM) withdraws the RCCA held by the mechanism. Mechanical failures are in the direction of insertion or immobility. Note that the operator can deliberately withdraw a single RCCA in a control or shutdown bank since this feature is necessary to retrieve an assembly should one drop accidentally.

A dropped RCCA or RCCA bank is detected by:

- Sudden drop in the core power level as seen by the Nuclear Instrumentation System
- Asymmetric power distribution as seen on out-of-core neutron detectors or core exit thermocouples
- Rod-at-bottom signal
- Rod deviation alarm
- Rod position indication

Dropping of a full-length RCCA is assumed to be initiated by a single electrical or mechanical failure that causes any number and combination of rods from the same group of a given control

bank to drop to the bottom of the core. The resulting negative reactivity insertion causes nuclear power to rapidly decrease. An increase in the hot channel factor can occur due to the skewed power distribution representative of a dropped rod configuration. For this event, it must be shown that the DNB design basis is met for the combination of power, hot channel factor, and other system conditions that exist following a dropped RCCA.

Misaligned RCCAs are detected by:

- Asymmetric power distribution as seen on out-of-core neutron detectors or core exit thermocouples
- Rod deviation alarm
- Rod position indicators

The resolution of the rod position indicator channel is ± 5 percent of span (± 7.2 inches). Any RCCA can deviate from its group by twice this distance (10 percent of span or 14.4 inches) and not cause power distributions exceeding design limits. The deviation alarm alerts the operator when any rod deviates from its group position by more than 5 percent of span. If the rod deviation alarm is not operable, the operator must take action as required by the plant Technical Specifications.

6.3.4.2 Input Parameters and Assumptions

For 1 or more dropped RCCA(s) in the same group, transient statepoints are generated generically and evaluated on a plant-specific, cycle-specific basis, to determine if the acceptance criteria are met. The statepoints, in the form of changes in key parameters from the initial values, are calculated based on the following conservative assumptions.

- This accident is analyzed with the RTDP, (Reference 3). Therefore, initial reactor power, pressure, and RCS average temperature are assumed to be at their nominal values. Uncertainties in initial conditions are included in the DNBR limit calculated using the referenced methodology.
- The transient statepoints are based on generic dropped rod analyses specifically performed to support elimination of turbine runback (on dropped rod) and deletion of the negative flux rate trip. The statepoint analysis bounds a dropped RCCA event for single or multiple dropped RCCAs from the same group of a given bank simulating rod withdrawal block (since automatic rod withdrawal had been physically disabled at IP2).

- A range of MTCs from 0 to -35 pcm/°F was analyzed, which bounds the limiting time in life.
- A range of negative reactivity insertions from 100 to 1000 pcm is assumed to simulate the dropped RCCA event.
- To provide a conservative analysis that minimizes the DNBR, the pressure-reducing functions of the automatic pressure control system are modeled. The pressure-reducing functions modeled are the pressurizer power-operated relief valves (PORVs) and spray.

6.3.4.3 Description of Analysis

Dropped RCCA(s) and RCCA Bank

The transient response following a dropped RCCA event was calculated using a detailed digital simulation of the plant. A dropped RCCA or dropped RCCA bank caused a step decrease in reactivity and the resulting core power generation was determined using the LOFTRAN computer code (Reference 6). The code simulates the neutron kinetics, RCS, pressurizer, pressurizer relief and safety valves, pressurizer spray, Rod Control System, steam generators, and steam generator safety valves. The code computes pertinent plant variables including temperatures, pressures, and power level. Since LOFTRAN employs a point neutron kinetics model, a dropped rod event was modeled as a negative reactivity insertion corresponding to the reactivity worth of the dropped RCCA(s), regardless of the actual configuration of the rod(s) that drop.

For the evaluation of the dropped RCCA event, generic transient statepoints designed to bound specific plant types were examined and found to be applicable (bounding) for IP2 at SPU conditions. The statepoints representing transient system conditions at the limiting point in the transient were calculated by the LOFTRAN code. No credit for any direct trip due to the dropped RCCA(s) was taken in the generic analysis. The generic analysis also assumed no automatic power reduction features (that is, turbine runback) were actuated by the dropped RCCA(s). The statepoints were provided for conditions that covered the range of reactivity parameters expected to occur during core life.

The statepoints and nuclear models specific for IP2 were used to obtain a hot channel factor consistent with the primary system conditions and reactor power. By incorporating the primary conditions from the transient and the hot channel factor from the nuclear analysis, the DNB design basis was shown to be met. The transient response, nuclear peaking factor analysis, and DNB design basis confirmation were performed in accordance with the dropped rod methodology described in WCAP-11394 (Reference 7).

Statically Misaligned RCCA

For the statically misaligned RCCA event, steady-state power distributions were analyzed at SPU power conditions (3216-MWt core) using appropriate nuclear physics computer codes. The VIPRE-01 (VIPRE-W) computer code (Reference 8) was used to determine the peaking factors that can meet the safety analysis limit DNBR. The analysis examined the case of the worst rod withdrawn from bank D inserted at the insertion limit with the reactor initially at full power. The analysis assumed this incident to occur at BOL since this resulted in the minimum feedback value (least negative) of the MTC. This assumption maximized the power rise and minimized the tendency of the large MTC (most negative) to flatten the power distribution.

6.3.4.4 Acceptance Criteria

Based on frequency of occurrence, a misaligned or dropped RCCA is considered a Condition II event as defined by the ANS. The limiting acceptance criteria for these events is that the critical heat flux should not be exceeded, as demonstrated by precluding DNB, and the peak linear heat generation rate should not exceed a value that could cause fuel centerline melt.

6.3.4.5 Results

Dropped RCCA(s) and RCCA Bank

Following 1 or more dropped RCCA(s) from the same group, a negative reactivity insertion resulted. The core was not adversely affected during this period since power is decreasing rapidly. Following the RCCA drop(s), the plant established a new equilibrium condition. Depending on the worth of the dropped RCCA(s), power can be reestablished by reactivity feedback.

When reactivity feedback did not offset the worth of the dropped RCCA(s), there was a cooldown condition until a low pressurizer-pressure reactor trip signal was reached. Figures 6.3-21 through 6.3-23 show a typical transient response at BOL conditions with a small negative MTC of $-5 \text{ pcm}/^{\circ}\text{F}$ for a dropped RCCA worth of 400 pcm.

When reactivity feedback was large enough to offset the worth of the dropped RCCA(s), reactor power was reestablished at a new equilibrium condition. Figures 6.3-24 through 6.3-26 show a typical transient response at EOL conditions with a large negative MTC of $-35 \text{ pcm}/^{\circ}\text{F}$ for a dropped RCCA worth of 400 pcm.

In all cases, the minimum DNBR remained above the safety analysis limit DNBR and the peak fuel centerline melt temperature criterion at the SPU condition was met.

Following plant stabilization, the operator can manually retrieve a dropped RCCA by following approved operating procedures.

Statically Misaligned RCCA

The most severe misalignment situations with respect to DNBR occurred at significant power levels. These situations arise from cases in which 1 RCCA is fully inserted or where bank D is fully inserted with 1 RCCA fully withdrawn. Multiple independent alarms, including a bank insertion limit alarm, alerts the operator well before the transient approaches the postulated conditions. The bank can be inserted to its insertion limit with any 1 assembly fully withdrawn without the DNBR falling below the safety analysis limit.

The insertion limits in the COLR may vary from time to time depending on several limiting criteria. The insertion limits on control bank D must be chosen to be above that position that meets the minimum DNBR and peaking factors. Detailed results will vary from cycle to cycle depending on fuel arrangements.

For this RCCA misalignment, with bank D to its full-power insertion limit and 1 RCCA fully withdrawn, DNBR did not fall below the safety analysis limit when analyzed at SPU conditions. The analysis of this case assumed that the initial reactor power, pressure, and RCS temperature were at their nominal values, with the increased radial peaking factor associated with the misaligned RCCA.

For RCCA misalignment with 1 RCCA fully inserted, the DNBR did not fall below the safety analysis limit when analyzed at SPU conditions. The analysis of this case assumed that the initial reactor power, pressure, and RCS temperatures were at their nominal values, with the increased radial peaking factor associated with the misaligned RCCA.

By meeting the DNBR limit for the RCCA misalignment incident there was no reduction in the ability of the primary coolant to remove heat from the fuel rod. The peak fuel temperature corresponded to a linear heat generation rate based on the radial peaking factor penalty associated with the misaligned RCCA and the limiting design axial power distribution. The resulting linear heat generation rate was well below that which would cause fuel melting.

6.3.4.6 Conclusions

Following a dropped RCCA(s) event the plant will return to a stabilized condition. Results of the analysis showed that a dropped RCCA event, with or without a reactor trip, did not adversely affect the uprated core since the DNBR remained above the limit for a range of dropped RCCA worths.

For all cases of any RCCA fully inserted, or bank D inserted to its rod insertion limits with any single RCCA in that bank fully withdrawn (statically misaligned RCCA), the DNBR remained greater than the safety analysis limit at uprated power conditions; thus, there was no reduction in the ability of the primary coolant to remove heat from the fuel rod. After identifying an RCCA group misalignment condition, the operator must take action as required by the plant Technical Specifications and operating instructions.

6.3.5 Chemical and Volume Control System Malfunction

6.3.5.1 Introduction

Reactivity can be added to the core with the Chemical and Volume Control System (CVCS) by feeding reactor makeup water into the RCS via the Reactor Makeup Control System. Boron dilution is a manual operation. A Boric Acid Blend System is provided to permit the operator to match the concentration of reactor coolant makeup water to that existing in the coolant at the time. The CVCS is designed to limit, even under various postulated failure modes, the potential rate of dilution to a value that, after indication through alarms and instrumentation, provides the operator sufficient time to correct the situation in a safe and orderly manner.

There is only a single, common source of dilution water to the blender from the primary water makeup system; inadvertent dilution can be readily terminated by isolating this single source. The operation of the primary water makeup pumps that take suction from the primary water storage tank (PWST) provides the non-borated supply of makeup water to the blender. The boric acid from the boric acid storage tank(s) is blended with the reactor makeup water in the blender, and the composition is determined by the preset flow rates of boric acid and reactor makeup water on the reactor makeup control. The operator must switch from the automatic makeup mode to the dilute mode and move the start-stop switch to start or, alternatively, the boric acid flow controller could be set to zero. Since these are deliberate actions, the possibility of inadvertent dilution is very small. For this dilution water to be added to the RCS, the charging pumps must be running in addition to the primary water makeup pumps. Also, any diluted water introduced into the volume control tank (VCT) must pass through the charging pumps to be added to the RCS.

Thus, the rate of addition of diluted water to the RCS from any source is limited to the capacity of the charging pumps. This addition rate is 294 gpm for all 3 charging pumps. This is the maximum delivery rate based on a pressure drop calculation comparing the pump curve with the system resistance curve. Normally, only 1 charging pump is operating while the others are on standby.

Information on the status of the reactor coolant makeup is continuously available to the operator. Lights are provided on the control board to indicate the operating condition of pumps in the CVCS. Alarms are actuated to warn the operator if boric acid or demineralized water flow rates deviate from preset values as a result of system malfunction. Postulated boron dilution events during refueling, startup, and power operation were considered in this analysis.

The CVCS malfunction event was analyzed for the refueling, startup, and power modes and addressed for the cold shutdown mode and hot shutdown mode (when $200^{\circ}\text{F} < T_{\text{avg}} \leq 350^{\circ}\text{F}$) by the Interim Operating Procedure.

6.3.5.1.1 Dilution during Refueling

In a dilution in the refueling mode, the operator has prompt and definite indication of any boron dilution from the audible count rate instrumentation. High count rate is alarmed in the reactor containment and the main control room. The count-rate increase is proportional to the multiplication factor.

In addition, there could be a source of water from Indian Point Unit 1 (IP1). Procedures call for isolation of that source should there be an unintended dilution.

6.3.5.1.2 Dilution during Startup

In this mode, the plant is being taken from one long-term mode of operation, hot standby, to another, power operation. Typically, the plant is maintained in the startup mode only for the purpose of startup testing at the beginning of each cycle. During this mode of operation, rod control is in manual. All normal actions required to change power level, either up or down, required operator initiation.

This mode of operation was a transitory operational mode in which the operator intentionally dilutes (borates) and withdraws control rods to take the plant critical. During this mode, the plant is in manual control with the operator required to maintain a high awareness of the plant status. For a normal approach to criticality, the operator has to manually initiate a limited dilution (boration) and subsequently manually withdraw the control rods, a process that takes several hours. The Technical Specifications require that the operator ensure that the reactor does not go critical with the control rods below the insertion limits. Once critical, the power escalation must be sufficiently slow to allow the operator to manually block the source range reactor trip nominally set at 2.3×10^5 CPS after receiving P-6 from the intermediate range. Too fast a power escalation (due to an unknown dilution) would result in reaching P-6 unexpectedly, leaving insufficient time to manually block the source range reactor trip. Failure to perform this manual action could result in a reactor trip and immediate shutdown of the reactor.

6.3.5.1.3 Dilution during Power Operation

In this mode, the plant could be operated in either automatic or manual rod control.

With the reactor in automatic rod control, the power and temperature increase from boron dilution results in insertion of the control rods and a decrease in the available shutdown margin. The rod insertion limit alarms (low and low-low settings) alert the operator that a dilution is in progress. The intent of the analysis in this mode is to show there is sufficient time to determine the cause of dilution, isolate the reactor water makeup source, and initiate boration before the available shutdown margin is lost (resulting in a return to critical condition).

With the reactor in manual control and no operator action taken to terminate the transient, the power and temperature rise would cause the reactor to reach the OTΔT trip setpoint resulting in a reactor trip. The boron dilution transient in this case would be essentially equivalent to an uncontrolled-RCCA-bank-withdrawal-at-power event. The maximum reactivity insertion rate for a boron dilution is conservatively estimated to be within the range of insertion rates analyzed in the RCCA bank withdrawal at power analysis. The intent of the analysis is to show there is sufficient time for the operator to determine the cause of dilution, isolate the reactor water makeup source, and initiate boration before the available shutdown margin is lost (resulting in a return to critical condition).

6.3.5.2 Input Parameters and Assumptions

6.3.5.2.1 Dilution during Refueling

Conditions assumed for the analysis were:

- Dilution flow is at the maximum capacity of the charging pumps—294 gpm.
- One RHR pump providing a minimum flow rate of 1000 gpm is normally running except during short time periods, as allowed by the Technical Specifications. A minimum active RCS water volume of 3257 ft³ is assumed. This corresponded to the active RCS volume while on RHR, and conservatively assumes an RCS vessel filled to mid-loop.
- The initial boron concentration is assumed to be 2050 ppm.
- The critical boron concentration following reactor trip is assumed to be 1390 ppm, corresponding to all rods inserted and no xenon condition. The 660-ppm change from the initial condition noted above is a conservative minimum value.

6.3.5.2.2 Dilution during Cold Shutdown

Boron dilution while in cold shutdown is addressed by the Interim Operating Procedure.

6.3.5.2.3 Dilution during Hot Shutdown

Boron dilution while in the hot shutdown mode (when $200^{\circ}\text{F} < T_{\text{avg}} \leq 350^{\circ}\text{F}$) is addressed by the Interim Operating Procedure.

6.3.5.2.4 Dilution during Startup

Conditions assumed for the analysis are:

- Dilution flow is at the maximum capacity of the charging pumps—294 gpm.
- A minimum RCS water volume of 8567 ft³ is modeled. This corresponds to the active RCS volume taking into account 10-percent uniform SGTP minus the pressurizer and the reactor vessel upper head
- The initial boron concentration is assumed to be 1800 ppm, which is a conservative maximum value for the critical concentration at the condition of HZP, rods to insertion limits, and no xenon.
- The critical boron concentration following reactor trip is assumed to be 1550 ppm, corresponding to the HZP, all rods inserted (minus the most reactive RCCA), and no xenon condition. The 250-ppm change from the initial condition noted above is a conservative minimum value.

6.3.5.2.5 Dilution during Full-Power Operation

In this mode, the plant can be operated in either automatic or manual rod control. Conditions assumed for the analysis were:

- Dilution flow is at the maximum capacity of the charging pumps—294 gpm.
- A minimum RCS water volume of 8567 ft³ is modeled. This corresponds to the active RCS volume (with 10-percent uniform SGTP minus the pressurizer and reactor vessel upper head).

- The initial boron concentration is assumed to be 1800 ppm, which is a conservative maximum value for the critical concentration at the condition of HFP, rods to insertion limits, and no xenon.
- The critical boron concentration following reactor trip is assumed to be 1450 ppm, corresponding to the HZP, all rods inserted (minus the most reactive RCCA), and no xenon condition. The 350-ppm change from the initial condition noted above is a conservative minimum value.

6.3.5.3 Description of Analysis

To cover all phases of plant operation, boron dilution during refueling and power modes of operation were considered in this analysis.

Conservative values for necessary parameters were used, that is, high RCS critical boron concentrations, high boron worth, minimum shutdown margins, and lower than actual RCS volumes. These assumptions result in conservative determinations of the time available for operator or system response after detection of a dilution transient in progress.

Conservative analysis methods were used to analyze a CVCS malfunction that resulted in a decrease in boron concentration in the reactor coolant. Minimum reactor coolant volumes and maximum dilution flow rates were conservatively assumed for each case analyzed. The result was a logarithmic decrease in coolant boron concentration according to the equation:

$$dC_B/dt = - [Q_{in}/V] C_B$$

Where:

C_B = Boron concentration in the RCS

Q_{in} = Maximum dilution flow rate

V = Active volume in RCS

This equation could be solved for the time at which the core would become critical or all shutdown margin would be lost. The rate of reactivity insertion due to the dilution could be calculated from the dilution rate and the differential boron worth. The results of this analysis were conservative for all cases analyzed.

Dilution events during the cold shutdown mode and the hot shutdown mode (when $200^\circ\text{F} < T_{avg} \leq 350^\circ\text{F}$) were addressed by the Interim Operating Procedure. An

evaluation was performed that concluded that the Interim Operating Procedure remain conservative for the cold and hot shutdown modes, as defined above, for the SPU Program.

6.3.5.4 Acceptance Criteria

A CVCS malfunction is classified as an ANS Condition II event, a fault of moderate frequency. Criteria established for Condition II events are as follows.

- The critical heat flux should not be exceeded. This is ensured by demonstrating that the minimum DNBR does not go below the limit value at any time during the transient.
- Pressure in the RCS and MSS should be maintained below 110 percent of the design pressures.
- Fuel temperature and fuel clad strain limits should not be exceeded. The peak linear heat generation rate should not exceed a value that would cause fuel centerline melt.

This event was analyzed to ensure that there is sufficient time for mitigation of an inadvertent boron dilution prior to the complete loss of shutdown margin. A complete loss of plant shutdown margin results in a return of the core to the critical condition, causing an increase in the RCS temperature and heat flux. This could violate the safety analysis DNBR limit and challenge fuel and fuel cladding integrity. A complete loss of plant shutdown margin could also result in a return of the core to the critical condition, causing an increase in RCS pressure. This could challenge the pressure design limit for the RCS.

If the minimum allowable shutdown margin is shown not to be lost, the condition of the plant at any point in the transient is within the bounds of those calculated for other Condition II transients. By showing that the above criteria were met for those Condition II events, it can be concluded that they were also met for the boron dilution event. Operator action was relied upon to preclude a complete loss of plant shutdown margin.

6.3.5.5 Results

6.3.5.5.1 Dilution during Refueling

From initiation of the event, there were 31.65 minutes available for operator action prior to return to criticality.

6.3.5.5.2 Dilution during Startup

From initiation of the event, there were 24.28 minutes available for operator action prior to return to criticality.

6.3.5.5.3 Dilution during Full-Power Operation

From initiation of the event while in manual rod control, there were 32.36 minutes available for operator action prior to loss of shutdown margin (return to criticality).

From initiation of the event while in automatic rod control, there were 33.88 minutes available for operator action prior to loss of shutdown margin (return to criticality).

6.3.5.6 Conclusions

The results of this analysis show that in the event of an uncontrolled inadvertent boron dilution, there is sufficient time for operator action to mitigate the consequences of this event prior to a complete loss of shutdown margin.

6.3.6 Loss-of-External Electrical Load

6.3.6.1 Introduction

A major loss of load can result from either a loss-of-external electrical load or from a turbine trip. A loss-of-external electrical load can result from an abnormal variation in network frequency or other adverse network operating conditions. For either case, offsite power is available for the continued operation of plant components such as the RCPs. The case of loss of all non-emergency AC power is presented in subsection 6.3.8 of this document.

For a loss-of-external electrical load without subsequent turbine trip, no direct reactor trip signal would be generated, as the plant would be expected to trip from the reactor protection system if a safety limit were approached. A continued steam load of approximately 5 percent would exist after total loss-of-external electrical load because of the steam demand of plant auxiliaries.

For a turbine trip, the reactor would be tripped directly (unless below P-8 power) on a signal from the turbine auto stop oil pressure or turbine stop valves.

If the steam dump valves fail to open following a large loss of load, the steam generator safety valves can lift and the reactor can be tripped by the high-pressurizer pressure signal, high-pressurizer water level signal, or OTΔT signal. If feedwater flow is also lost, the reactor can be

tripped by a steam generator low-low water level signal. The steam generator shell-side pressure and reactor coolant temperatures will increase rapidly following a large loss of load. The pressurizer and steam generator safety valves are sized to protect the RCS and steam generators against overpressure for all load losses without assuming operation of the steam dump system, pressurizer spray, pressurizer PORVs, automatic rod control, or direct reactor trip on turbine trip.

The PSV capacity is sized based on a complete loss of heat sink with the plant initially operating at the maximum calculated turbine load along with operation of the steam generator safety valves. The pressurizer and steam generator safety valves are then able to maintain the RCS and MSS pressures within 110 percent of the corresponding design pressure without a direct reactor trip on turbine trip.

6.3.6.2 Input Parameters and Assumptions

The loss-of-external electrical load/turbine trip accident is analyzed for two specific cases:

- Maximum RCS and secondary side pressures
- Minimum DNBR

The major assumptions used in the analyses are summarized below.

Initial Operating Conditions

The peak pressure case without pressure control is analyzed using the STDP. Initial reactor power and RCS temperatures are assumed to be at their nominal values plus uncertainties. Initial RCS pressure is assumed to be at its nominal value minus uncertainties. The analysis models thermal design flow (322,800 gpm).

The minimum DNBR case with pressure control is analyzed using the RTDP (Reference 3). Initial reactor power, pressure, and RCS average temperature are assumed to be at their nominal values. Uncertainties in initial conditions are included in the DNBR limit. Minimum measured flow was modeled.

Reactivity Coefficients

Minimum reactivity feedback (BOL) conditions are conservatively assumed for both cases. The analysis is performed at full-power conditions assuming an MTC of 0 pcm/°F. Least negative Doppler coefficients are also assumed.

Reactor Control

From the standpoint of the maximum pressures and minimum DNBR attained, it is conservative to assume that the reactor is in manual rod control. If the reactor were in automatic rod control, the control rod banks would insert prior to trip and reduce the severity of the transient.

Pressurizer Spray and PORVs

The pressurizer PORVs and pressurizer spray portion of the automatic pressure control system are assumed in the minimum DNBR case since each serves to limit the RCS pressure increase, which is conservative for the DNBR calculation. In the peak pressure case, the pressurizer PORVs and spray are not assumed. In each case, safety valves are assumed operable with a capacity of 408,000 lbm/hr per valve for three valves.

Feedwater Flow

Main feedwater flow to the steam generators is assumed to be lost at the time of turbine trip. No credit is taken for AFW flow; however, eventually AFW flow would be initiated and a stabilized plant condition would be reached.

Reactor Trip

Reactor trip is actuated by the first RPS trip setpoint reached. Trip signals are expected due to high-pressurizer pressure and OTΔT.

Steam Release

No credit is taken for operating the Steam Dump System or steam generator ARVs. This assumption maximizes secondary pressure.

6.3.6.3 Description of Analysis

For the loss-of-external-electrical load/turbine trip event, the behavior of the unit was analyzed for a complete loss of steam load from full power without a direct reactor trip. This assumption was made to show the adequacy of the pressure-relieving devices and to demonstrate core protection margins by delaying reactor trip until conditions in the RCS resulted in a trip due to other signals. Thus, the analysis assumed a worst-case transient. In addition, no credit was taken for steam dump. Main feedwater flow was terminated at the time of turbine trip, with no credit taken for AFW (except for long-term recovery) to mitigate the consequences of the transient.

A detailed analysis using the RETRAN (Reference 1) computer code was performed to determine the plant transient conditions following a total loss of load. The code modeled the core neutron kinetics, RCS including natural circulation, pressurizer, pressurizer PORVs and spray, steam generators, MSSVs, and the Auxiliary Feedwater System (AFWS); and computed pertinent variables, including pressurizer pressure, steam generator pressure, steam generator mass, and reactor coolant average temperature.

6.3.6.4 Acceptance Criteria

Based on its frequency of occurrence, the loss-of-external-electrical load/turbine-trip accident is considered a Condition II event as defined by the ANS. The criteria are as follows.

- Pressure in the RCS and MSS should remain below 110 percent of the design values.
- Fuel cladding integrity should be maintained by ensuring that the minimum DNBR remains above the 95/95 DNBR limit for PWRs.
- An incident of moderate frequency should not generate a more serious plant condition without other faults occurring independently. This criterion is met by ensuring that the pressurizer does not reach a water-solid condition. By precluding a water-solid condition, the potential for damage to the PSVs due to water relief is precluded and the RCS pressure boundary is uncompromised (that is, the Condition II event will not progress into a Condition III or IV type event).
- An incident of moderate frequency in combination with any single active component failure, or single operator error, should be considered an event for which an estimate of the number of potential fuel failures should be provided for radiological dose calculations. For such accidents, fuel failure must be assumed for all rods for which the DNBR falls below those values cited above for cladding integrity unless it can be shown, based on an acceptable fuel damage model, that fewer failures occur. There should be no loss of function of any fission product barrier other than the fuel cladding.

6.3.6.5 Results

The calculated sequence of events for the loss-of-external-electrical load/turbine-trip cases are presented in Table 6.3-6.

Peak Pressure Case

The transient responses for the total loss of steam load from full power are shown in Figures 6.3-27 and 6.3-28. No credit was taken for the pressurizer spray, pressurizer PORVs, or for the steam dump. The reactor was tripped by the high-pressurizer pressure trip channel.

The PSVs were actuated and the primary system pressure remained below the 110-percent design value. The steam generator safety valves maintained the secondary side steam pressure below 110 percent of the steam generator shell design pressure.

Minimum DNBR Case

The transient responses for the total loss of steam load from full power are shown in Figures 6.3-29 and 6.3-30. Full credit was taken for the pressurizer spray and pressurizer PORVs. No credit was taken for the steam dump. The reactor was tripped by the high-pressurizer pressure reactor trip channel. The minimum DNBR remained well above the limit value.

6.3.6.6 Conclusions

The results of this analysis showed that the plant design is such that a total loss-of-external-electrical-load transient without a direct reactor trip presented no hazard to the integrity of the RCS or the MSS at SPU conditions. All of the applicable acceptance criteria were met. The minimum DNBR for each case was greater than the safety analysis limit value. The peak primary and secondary system pressures remained below 110 percent of design at all times. The protection features presented in subsection 6.3.6.3 provided mitigation of the loss-of-external-electrical load/turbine trip transient so that the above criteria were satisfied.

6.3.7 Loss-of-Normal Feedwater

6.3.7.1 Introduction

A LONF (from pump failures, valve malfunctions, or LOAC) results in a reduction in capability of the secondary system to remove the heat generated in the reactor core. If the reactor were not tripped during this accident, core damage would possibly occur as a result of the loss-of-heat sink while at power. If an alternative supply of feedwater is not supplied to the plant, residual heat following a reactor trip may heat the primary system water to the point where water relief from the pressurizer could occur. A significant loss of water from the RCS could lead to core uncover and subsequent core damage. Since a reactor trip occurs well before the steam generator heat transfer capability is reduced, the primary system conditions never approach those that would result in a DNB condition.

The loss of normal feedwater that occurs as a result of the LOAC power is discussed in subsection 6.3.8 of this report.

The following events occur after the reactor trip for the LONF resulting from main feedwater pump failures or valve malfunctions:

- As steam system pressure rises following the trip, the steam generator ARVs can be automatically opened. Steam dump to the condenser is assumed not available. If the steam generator ARVs are not available, the MSSVs can lift to dissipate the sensible heat of the fuel and coolant plus the residual decay heat produced in the reactor core.
- As the no-load temperature is approached, the steam generators ARVs (or the MSSVs, if the steam generators ARVs are not available) are used to dissipate the residual decay heat and RCP heat and to maintain the plant at the hot standby condition.

Following the occurrence of a loss-of-normal feedwater, the reactor can be tripped on any of the following RPS trip signals:

- Low-low steam generator water level
- OTΔT
- High-pressurizer pressure
- High-pressurizer water level
- RCP undervoltage (if coincident with a LOOP signal)
- Steam flow-feedwater flow mismatch in coincidence with low water level in any steam generator

AFW is supplied by the actuation of 2 motor-driven AFW pumps (MDAFWPs), which are initiated by any of the following signals.

- Low-low water level in any steam generator
- Automatic trip (not manual) of any main feedwater pump turbine
- Any safety injection (SI) signal
- Manual actuation
- LOOP concurrent with unit trip

In addition, 1 turbine-driven AFW pump (TDAFWP) starts on any of the following actuation signals, although no automatic delivery of water to the steam generators occurs (the TDAFWP is automatically started, but must be manually aligned by the operator to allow delivery of AFW flow to the steam generators).

- Low-low water level in any 2 steam generators
- Loss of offsite power (LOOP) concurrent with unit trip and no safety injection signal
- Manual actuation

The MDAFWPs are powered by the emergency diesel generators (EDGs). The pumps take suction from the condensate storage tank (CST) for delivery to the steam generators. Each MDAFWP is designed to supply the minimum required flow within 60 seconds of the initiating signal. The TDAFWP is valved-out during normal operation. Therefore, although the TDAFWP is automatically actuated, this pump is not available to deliver flow to the steam generators until operator action is taken to align the TDAFWP.

Backup in equipment and control logic is provided to ensure that reactor trip and automatic AFW flow will occur following any LONF, including that followed by a LOAC. The analysis showed that following a LONF, the AFWS is capable of removing the stored and residual heat plus RCP waste heat, thus preventing overpressurization of the RCS and the steam generator secondary side, water relief from the pressurizer, and uncover of the reactor core.

6.3.7.2 Input Parameters and Assumptions

The analysis was performed for IP2 at SPU conditions. The major assumptions used in this analysis were as follows.

- The plant is initially operating at 102 percent of the uprated NSSS power (3230 MWt) and bounds a nominal pump heat of 14 MWt.
- The RCPs are assumed to operate continuously throughout the transient providing a constant reactor coolant volumetric flow equal to the thermal design flow (TDF).
- Cases are analyzed assuming initial HFP T_{avg} at the upper and lower ends of the SPU operating range with uncertainty applied in both the positive and negative direction. The vessel average temperature assumed at the upper end of the range is 572°F with an uncertainty of $\pm 7.5^\circ\text{F}$. The average temperature assumed at the lower end of the range is 549°F with an uncertainty of $\pm 7.5^\circ\text{F}$.
- Initial pressurizer pressure is assumed to be 2250 psi with an uncertainty of +28/-37 psi. Cases are considered with the pressure uncertainty applied in both the positive and negative direction to conservatively bound potential operating conditions.
- Cases are analyzed assuming initial feedwater temperatures at the upper and lower ends of the uprated operating feedwater temperature window (436.2 and 390°F, respectively).
- Reactor trip occurs on steam generator low-low water level at 0-percent narrow range span (NRS).

- The worst single failure modeled in the analysis is the loss of 1 of the 2 MDAFWPs. This results in the availability of only 1 MDAFWP automatically supplying a minimum total AFW flow of 380 gpm, distributed equally between 2 of the 4 steam generators. Additional flow from the second MDAFWP or TDAFWP is assumed to be available only following operator action to start the second MDAFWP or to align the TDAFWP. This operator action is assumed to provide an additional 380 gpm of AFW flow distributed equally to the other 2 steam generators not receiving AFW automatically, and is assumed to occur at 10 minutes after the reactor trip due to a low-low steam generator water level signal.
- The automatic AFW flow is assumed to be initiated 60 seconds following a low-low steam generator water level signal.
- The pressurizer spray, PORVs, and heaters are assumed to be operable to maximize the pressurizer water volume. Note that these control systems are not required for event mitigation since the PSVs alone would prevent the RCS pressure from exceeding the design limit during this transient.
- Secondary system steam relief is achieved through the MSSVs, which are modeled assuming a +3-percent lift setpoint tolerance, a 5-psi ramp for the valve to pop open, and a pressure difference between each steam generator and the safety valves of approximately 20 psi at full relief flow. Steam relief through the steam generator ARVs or condenser dump valves is assumed unavailable.
- A conservative core residual heat generation based upon long-term operation at the initial power level preceding the trip is assumed. This core residual heat generation model is based on the 1979 version of ANS 5.1 (Reference 9). ANS/ANS-5.1-1979 with 2σ is a conservative representation of the decay energy release rates.
- Analysis with both minimum (0 percent) and maximum (10 percent) SGTP is performed to conservatively bound potential operating conditions.
- A maximum AFW enthalpy of 90.77 Btu/lbm is conservatively assumed. An AFW line purge volume of 268.8 ft³ is modeled.

6.3.7.3 Description of Analysis

A detailed analysis using the RETRAN (Reference 1) computer code was performed to determine the plant transient following a loss-of-normal feedwater. The code simulates the core neutron kinetics, RCS, pressurizer, pressurizer PORVs and safety valves, pressurizer heaters

and spray, steam generators, MSSVs, and the AFWS, and computed pertinent variables, including pressurizer pressure, pressurizer water level, steam generator mass, and reactor coolant average temperature.

6.3.7.4 Acceptance Criteria

Based on its frequency of occurrence, the LONF accident is considered a Condition II event as defined by the ANS. The following items summarize the acceptance criteria associated with this event.

- The critical heat flux should not be exceeded. This is demonstrated by ensuring that the applicable safety analysis DNBR limit is met. With respect to DNB, the LONF accident was bounded by the loss-of-load accident described in subsection 6.3.6.
- Pressure in the RCS and MSS should be maintained below 110 percent of the design pressures. With respect to RCS and MSS overpressurization, the LONF accident was bounded by the loss-of-load accident reported in subsection 6.3.6.
- An incident of moderate frequency should not generate a more serious plant condition without other faults occurring independently. This criterion is met by ensuring that the pressurizer does not reach a water-solid condition. By precluding a water-solid condition in the pressurizer, the potential for damage to the pressurizer PORVs and safety valves due to water relief is precluded and the RCS pressure boundary is not compromised (that is, the Condition II event will not progress into a Condition III or IV type event).

6.3.7.5 Results

The calculated sequence of events for this accident is listed in Table 6.3-7. Figures 6.3-31 through 6.3-39 present the transient response of plant conditions and parameters of interest following a LONF with the assumptions listed in subsection 6.3.7.2 of this document.

Following the reactor and turbine trip from full load, the water level in the steam generators will fall due to reduction of the steam generator void fraction and because steam flow through the MSSVs continued to dissipate the stored and generated heat. Approximately 1 minute following the initiation of the low-low steam generator water level trip, 1 MDAFWP started automatically, consequently reducing the rate at which the steam generator water level decreased in the 2 steam generators receiving automatic AFW flow. Operator action to start the second MDAFWP or to align the TDAFWP at 10 minutes after reactor trip on a low-low steam generator water level signal was assumed to deliver additional AFW flow to the 2 steam generators not already receiving AFW and the plant was brought to a stable condition.

The pressurizer never reached a water-solid condition (see Figure 6.3-32). Hence, no water relief from the pressurizer occurred. The peak RCS and MSS pressures remained below the applicable design limits throughout the transient.

Since the plant was tripped well before the steam generator heat transfer capability was reduced, the primary system variables never approached a DNB condition.

6.3.7.6 Conclusions

With respect to DNB, the LONF accident was bounded by the loss-of-load accident (see subsection 6.3.6), which demonstrated that the minimum DNBR remained greater than the safety analysis limit.

The results of the analysis showed that the pressurizer did not reach a water-solid condition and that the applicable RCS and MSS pressure limits were met. Therefore, the LONF event did not adversely affect the core, RCS, or MSS.

6.3.8 LOAC to the Station Auxiliaries

6.3.8.1 Introduction

A complete loss of non-emergency AC power can result in the loss-of-all-power to the plant auxiliaries, such as the RCPs or condensate pumps. The loss-of-power to the condensate pumps results in a LONF. The loss of power may be caused by a complete loss-of-the-offsite grid accompanied by a turbine generator trip at the station, or by a loss-of-the-onsite AC distribution system.

The first few seconds of the transient would be almost identical to the complete loss-of-flow accident presented later in subsection 6.3.13, in which the pump coastdown inertia along with the reactor trip prevents reaching the DNBR limit. After the trip, decay heat removal will be accommodated by the AFWS. This portion of the transient would be similar to that presented in subsection 6.3.7 for the LONF event.

Following a LOAC with turbine and reactor trips, the sequence described below will occur.

- Plant vital instruments are supplied from emergency DC power sources.
- As the steam system pressure rises following the trip, the steam generator ARVs can be automatically opened to atmosphere. The condenser is assumed not available for steam dump because of the loss of the circulating water pumps. If the steam generator

ARVs are not available, the MSSVs can lift to dissipate the sensible heat of the fuel and coolant, plus the residual decay heat produced in the reactor.

- As the no-load temperature is approached, the steam generator ARVs (or the MSSVs, if the steam generator ARVs are not available) are used to dissipate the residual decay heat and to maintain the plant at the hot standby condition.
- The EDGs will start on a loss of voltage on the plant emergency buses and begin to supply plant vital loads.

The AFWS is started automatically as discussed previously in subsection 6.3.7 for the LONF analysis. The AFWS comprises 2 MDAFWPs and 1 TDAFWP. The TDAFWP uses steam from the secondary system and exhausts the steam to the atmosphere. The 2 MDAFWPs are supplied by power from the EDGs and take suction directly from the CST for delivery to the steam generators. Each MDAFWP is designed to supply the minimum required flow within 60 seconds of the initiating signal, even if a loss of all non-emergency AC power occurs simultaneously with a LONF. The TDAFWP is started automatically. However, the TDAFWP needs to be manually aligned before AFW flow can be delivered to the steam generators.

Following the loss of power to the RCPs, the RCPs coast down and the removal of residual decay heat is provided by natural circulation in the RCS, supported by AFW flow to the secondary system. Demonstrating that acceptable results can be obtained for this event shows that the natural circulation flow in the RCS is adequate to remove decay heat from the core.

The analysis of the LOAC event is performed to demonstrate that natural circulation in the RCS, along with the AFWS, is capable of removing the stored and residual decay heat from the core, and consequently preventing RCS or MSS overpressurization, water relief from the pressurizer, and uncover of the reactor core.

6.3.8.2 Input Parameters and Assumptions

The analysis was performed for IP2 at SPU conditions. The major assumptions used in this analysis were as follows.

- The plant is initially operating at 102 percent of the uprated NSSS power (3230 MWt). A nominal RCP heat of 14 MWt is assumed for the duration of the event.
- The initiating event is a loss of all non-emergency AC power that resulted in a loss of power to the condensate pumps. The loss of the condensate pumps results in a LONF.

- The RCPs are conservatively assumed to operate until the time of reactor trip, providing a constant reactor coolant volumetric flow equal to the TDF value. This assumption maximizes the amount of stored energy in the RCS. The loss of power to the RCPs is not assumed to occur until after the start of rod motion following the reactor trip on a low-low steam generator water level condition.
- No credit is taken for the immediate insertion of the control rods due to the LOAC to the station auxiliaries.
- Cases are analyzed assuming initial HFP T_{avg} at the upper and lower ends of the SPU operating range with uncertainty applied in both the positive and negative direction. The vessel average temperature assumed at the upper end of the range is 572°F with an uncertainty of $\pm 7.5^\circ\text{F}$. The average temperature assumed at the lower end of the range is 549°F with an uncertainty of $\pm 7.5^\circ\text{F}$.
- Initial pressurizer pressure is assumed to be 2250 psia with an uncertainty of +28/-37 psi. Cases are considered with the pressure uncertainty applied in both the positive and negative direction to conservatively bound potential operating conditions.
- Cases are analyzed assuming initial feedwater temperatures at the upper and lower ends of the updated operating feedwater temperature window (436.2 and 390°F, respectively).
- Reactor trip occurs on steam generator low-low water level at 0 percent of NRS.
- The worst single failure modeled in the analysis is the loss of 1 of the 2 MDAFWPs. This results in the availability of only 1 MDAFWP automatically supplying a minimum total AFW flow of 380 gpm, distributed equally between 2 of the 4 steam generators. Additional flow from the second MDAFWP or TDAFWP is assumed to be available only following operator action to start the second MDAFWP or to align the TDAFWP. This operator action is assumed to provide an additional 380 gpm of AFW flow distributed equally to the other 2 steam generators not receiving AFW automatically, and is assumed to occur at 10 minutes after the reactor trip due to a low-low steam generator water level signal.
- The automatic AFW flow is assumed to be initiated 60 seconds after a low-low steam generator water level signal.
- The pressurizer spray, PORVs, and heaters are assumed to be operable to maximize the pressurizer water volume. Note that these control systems are not required for event

mitigation since the PSVs alone would prevent the RCS pressure from exceeding the design limit during this transient.

- Secondary system steam relief is achieved through the MSSVs, which are modeled assuming a +3 percent lift setpoint tolerance, a 5-psi ramp for the valve to pop open, and a pressure difference between each steam generator and the safety valves of approximately 20 psi at full relief flow. Steam relief through the steam generator ARVs or condenser dump valves is assumed unavailable.
- A conservative core residual heat generation based upon long-term operation at the initial power level preceding the trip is assumed. This core residual heat generation model is based on the 1979 version of ANS 5.1 (Reference 9). ANSI/ANS-5.1-1979 with 2σ is a conservative representation of the decay energy release rates.
- Analysis with both minimum (0 percent) and maximum (10 percent) SGTP is performed to conservatively bound potential operating conditions.
- A maximum AFW enthalpy of 90.77 Btu/lbm is conservatively assumed. An AFW line purge volume of 268.8 ft³ is modeled.

6.3.8.3 Description of Analysis

A detailed analysis using the RETRAN (Reference 1) computer code was performed to determine the plant transient following a LOAC. The code simulates the core neutron kinetics, RCS including natural circulation, pressurizer, pressurizer PORVs and safety valves, pressurizer heaters and spray, steam generators, MSSVs, and the AFWS. RETRAN then computed pertinent variables, including pressurizer pressure, pressurizer water level, steam generator mass, and reactor coolant average temperature.

6.3.8.4 Acceptance Criteria

Based on its frequency of occurrence, the loss-of-non-emergency-AC-power accident is considered a Condition II event as defined by the ANS. The following items summarize the acceptance criteria associated with this event.

- The critical heat flux should not be exceeded. This is demonstrated by ensuring that the applicable safety analysis DNBR limit is met. With respect to DNB, the loss-of-non-emergency-AC-power accident is bounded by the complete loss-of-flow accident reported in subsection 6.3.13.

- Pressure in the RCS and MSS should be maintained below 110 percent of the design pressures. With respect to RCS and MSS overpressurization, the loss-of-non-emergency-AC-power accident is bounded by the loss-of-load accident reported earlier in subsection 6.3.6.
- An incident of moderate frequency should not generate a more serious plant condition without other faults occurring independently. This criterion is met by ensuring that the pressurizer does not reach a water-solid condition. By precluding a water-solid condition in the pressurizer, the potential for damage to the pressurizer PORVs and safety valves due to water relief is precluded and the RCS pressure boundary is not compromised (that is, the Condition II event will not progress into a Condition III or IV type event).

6.3.8.5 Results

Figures 6.3-40 through 6.3-48 present the transient response of plant conditions and parameters of interest following a loss-of-non-emergency-AC-power with the assumptions listed earlier in subsection 6.3.8.2. The calculated sequence of events for this accident is listed in Table 6.3-8.

During the first few seconds after the loss-of-non-emergency-AC power to the RCPs, the RCS flow transient closely resembled the complete loss-of-flow incident, where core damage due to rapidly increasing core temperature is prevented by reactor trip, which, for a loss-of-non-emergency-AC-power event, is on a low-low steam generator water level signal. After reactor trip, stored and residual decay heat must be removed to prevent damage to the core and the RCS and MSS. The RETRAN code results showed that natural circulation and the AFW flow available were sufficient to provide adequate core decay heat removal following reactor trip and RCP coastdown.

The pressurizer never reached a water-solid condition (see Figure 6.3-41). Hence, no water relief from the pressurizer occurred. The peak RCS and MSS pressures remained below the applicable design limits throughout the transient.

6.3.8.6 Conclusions

With respect to DNB, the loss-of-non-emergency-AC-power event was bounded by the complete loss-of-flow event (see subsection 6.3.13), demonstrating that the minimum DNBR remained above the safety analysis limit.

The results of the analysis showed that the pressurizer did not reach a water-solid condition and that the applicable RCS and MSS pressure limits were met. Therefore, the LOOP did not adversely affect the core, the RCS, or the MSS.

6.3.9 Excessive Heat Removal Due to Feedwater System Malfunction

6.3.9.1 Introduction

Reductions in feedwater temperature or excessive feedwater additions are a means of increasing core power above full power. Such transients are attenuated by the thermal capacity of the RCS and the secondary side of the plant. The overpower/overtemperature protection functions (neutron high flux, OT Δ T, and OP Δ T trips) prevent any power increase that could lead to a DNBR that is less than the limit value.

An example of excessive feedwater flow would be a full opening of 1 feedwater control valve due to a Feedwater Control System malfunction or an operator error. At power, this excess flow causes a greater load demand on the RCS due to increased subcooling in the steam generator. With the plant at no-load conditions, the addition of cold feedwater can cause a decrease in RCS temperature and thus, a reactivity insertion due to the effects of the negative MTC of reactivity. Continuous excessive feedwater addition is prevented by the steam generator high-high water level trip.

A second example of excess heat removal is the transient associated with failure of the low-pressure heaters' bypass valve resulting in an immediate reduction in feedwater temperature. At power, this increased subcooling will create a greater load demand on the RCS. However, the feedwater bypass valve has been retired-in-place. Thus, this event is no longer credible and was not considered here.

6.3.9.2 Input Parameters and Assumptions

The reactivity insertion rate following a feedwater system malfunction, attributed to the cooldown of the RCS, was calculated with the following assumptions:

- This accident is analyzed with the RTDP as described in WCAP-11397 (Reference 3). Initial reactor power, RCS pressure, and RCS temperature are assumed to be at their nominal values, adjusted to account for any applicable measurement biases, consistent with steady-state, full-power operation. Minimum measured flow (MMF) is modeled. Uncertainties in initial conditions are included in the DNBR limit as described in WCAP-11397 (Reference 3).

- The analyses are done at the updated NSSS power level of 3230 MWt.
- For the feedwater control valve accident at full-power conditions that results in an increase in feedwater flow to 1 steam generator, 1 feedwater control valve is assumed to malfunction, resulting in a step increase to 130 percent of the nominal full-power feedwater flow to 1 steam generator.
- The increase in feedwater flow rate results in a decrease in the feedwater temperature due to the reduced efficiency of the feedwater heaters. For the HFP cases, a 20°F decrease in the feedwater temperature is assumed to occur coincident with the feedwater flow increase.
- For the feedwater control valve accident at zero-load conditions that results in an increase in feedwater flow to 1 steam generator, 1 feedwater control valve is assumed to malfunction, resulting in a step increase to 210 percent of the nominal full-load value for one steam generator.
- For cases at zero-load conditions, feedwater temperature is assumed to be 100°F.
- The initial water level in all the steam generators is a conservatively low level.
- No credit is taken for the heat capacity of the RCS and steam generator metal mass in attenuating the resulting plant cooldown.
- The feedwater flow resulting from a fully open control valve is terminated by the steam generator high-high water level signal that closes all feedwater main control and feedwater control-bypass valves, indirectly closes all feedwater pump discharge valves, and trips the main feedwater pumps and turbine generator.

The RPS features, including power-range high neutron flux, OTΔT, and turbine trip on high-high steam generator water level, are available to provide mitigation of the feedwater system malfunction transient.

Normal reactor control systems and engineered safety systems (for example, SI) are not assumed to function. The RPS can actuate to trip the reactor due to an overpower condition. No single active failure in any system or component required for mitigation will adversely affect the consequences of this event.

6.3.9.3 Description of Analysis

The excessive heat removal due to a feedwater system malfunction transient was analyzed with the RETRAN (Reference 1) computer code. This code simulates a multi-loop system, neutron kinetics, the pressurizer, pressurizer relief and safety valves, pressurizer spray, steam generator, and main steam safety valves. The code computed pertinent plant variables including temperatures, pressures, and power level.

The excessive-feedwater-flow event assumed an accidental opening of 1 feedwater control valve with the reactor at both full- and zero-power conditions with both automatic and manual rod control. Both the automatic and manual rod control cases assumed a conservatively large moderator density coefficient characteristic of EOL conditions.

6.3.9.4 Acceptance Criteria

Based on its frequency of occurrence, the feedwater-system-malfunction event is considered a Condition II event as defined by the ANS. Even though DNB is the primary concern in the analysis of the feedwater malfunction event, the following three items summarize the criteria associated with this transient:

- The critical heat flux should not be exceeded. This is met by demonstrating that the minimum DNBR does not go below the limit value at any time during the transient.
- Pressure in the RCS and MSS should be maintained below 110 percent of the design pressures.
- The peak linear heat generation rate should not exceed a value that would cause fuel centerline melt.

6.3.9.5 Results

The excessive-feedwater-flow full-power case with automatic rod control yielded results that were nearly identical to the case assuming manual rod control. Considering cases with and without automatic rod control and presenting the more limiting results demonstrated that the Rod Control System was not required to function for this event. A turbine trip, which resulted in a reactor trip, was actuated when the steam generator water level in the affected steam generator reached the high-high water level setpoint. The results presented were from the case assuming that the automatic Rod Control System was not operable.

The case initiated at HZP conditions with manual rod control was less limiting than the HZP steamline break analysis. Therefore, the results of the HZP case were not presented.

For all cases of excessive feedwater flow, continuous addition of cold feedwater was prevented by automatic closure of all feedwater control and isolation valves, closure of all feedwater bypass valves, a trip of the feedwater pumps, and a turbine trip on high-high steam generator water level. In addition, the feedwater pump discharge isolation valves will automatically close upon receipt of the feedwater pump trip signal.

Following turbine trip, the reactor will automatically be tripped, either directly due to the turbine trip or due to 1 of the reactor trip signals discussed in subsection 6.3.6 (loss-of-external-electrical-load and/or turbine trip). If the reactor was in automatic rod control, the control rods would be inserted at the maximum rate following the turbine trip, and the resulting transient would not be limiting in terms of peak RCS pressure.

The effects of the RTDP methodology, including Rod Control System response characteristics were incorporated into the analysis. Table 6.3-9 shows the time sequence of events for the HFP feedwater malfunction transient. Figures 6.3-49, 6.3-50 and 6.3-51 show transient responses for various system parameters during a feedwater system malfunction initiated from HFP conditions with manual rod control.

6.3.9.6 Conclusions

For the excessive-feedwater-addition-at-power transient, the results showed that the DNBRs encountered were above the limit value, hence, no fuel damage was predicted.

The protection features presented previously in subsection 6.3.9.2 provided mitigation of the feedwater-system-malfunction transient so that the above criteria were satisfied.

6.3.10 Excessive Load Increase Incident

6.3.10.1 Introduction

An excessive load increase incident is defined as a rapid increase in the steam flow that causes a power mismatch between the reactor core power and the steam generator load demand. The RCS is designed to accommodate a 10-percent step-load increase or a 5-percent per minute ramp-load increase in the range of 15 to 100 percent of full power, taking credit for all control systems in automatic. Any loading rate in excess of these values can cause a reactor trip actuated by the RPS.

This accident could result from either an administrative violation such as excessive loading by the operator or an equipment malfunction in the steam dump control or turbine speed control. For excessive loading by the operator or by system demand, the turbine load limiter keeps the maximum turbine load at 100-percent rated load.

During power operation, steam dump to the condenser is controlled by comparing the RCS temperature (median T_{avg}) to a reference temperature based on turbine power, where a high temperature difference in conjunction with a loss of load or a turbine trip indicates a need for steam dump. A single controller or control signal malfunction does not cause steam dump valves to open. Interlocks are provided to block the opening of the valves unless a large turbine load decrease or a turbine trip has occurred. In addition, the reference temperature and loss-of-load signals are developed by independent sensors.

Protection against an excessive load increase accident is provided by the following RPS signals:

- OPΔT
- OTΔT
- Power range high neutron flux
- Low-pressurizer pressure

6.3.10.2 Input Parameters and Assumptions

The analysis includes the following conservative assumptions:

- This event is analyzed with the RTDP (Reference 3). Initial reactor power and RCS pressure and temperature are assumed to be at their nominal values. Uncertainties in initial conditions are included in the limit DNBR as described in WCAP-11397-A (Reference 3).
- The evaluation is performed for a step-load increase of 10 percent steam flow from 100 percent RTP.
- The excessive load increase event is analyzed for both BOL (minimum reactivity feedback) and EOL (maximum reactivity feedback) conditions.

6.3.10.3 Description of Analysis

Two cases were considered to demonstrate that the fuel cladding integrity will not be adversely affected following a 10-percent step-load increase from rated load. This was shown by demonstrating that the minimum DNBR would not go below the safety analysis limit value.

- Manually controlled reactor with BOL (minimum moderator) reactivity feedback
- Manually controlled reactor with EOL (maximum moderator) reactivity feedback

At BOL minimum moderator feedback conditions, the core had the least-negative MTC of reactivity and the least-negative Doppler-only power coefficient curve, and therefore, the least-inherent transient response capability. Since a positive MTC would provide a transient benefit, a zero MTC was evaluated for the minimum feedback conditions. For the EOL maximum moderator feedback conditions, the MTC of reactivity had its most-negative value and the most-negative Doppler-only power coefficient curve. This resulted in the largest amount of reactivity feedback due to changes in coolant temperature.

The effect of this transient on the minimum DNBR was evaluated by applying conservatively large deviations to the initial conditions of core power, average coolant temperature, and pressurizer pressure at the normal full-power operating conditions to generate a limiting set of statepoints. These deviations bound the variations that could occur as a result of an excessive load increase accident and were only applied in the direction that had the most adverse effect on the DNB ratio; namely increased power, coolant temperature, and decreased pressure. No credit was taken for the decrease in coolant temperature and no reactor trip is assumed.

The reactor condition statepoints (temperature, pressure, and power) were compared to the conditions corresponding to operation at the safety analysis DNB limit.

Normal reactor control systems and engineered safety systems were not required to function. A conservative limit on the turbine valve opening was assumed. The analysis did not take credit for pressurizer heaters.

The RPS was assumed to be operable. However, reactor trip was not encountered for most cases due to the error allowances assumed in the setpoints. No single active failure in any system or component required for mitigation will adversely affect the consequences of this accident.

6.3.10.4 Acceptance Criteria

Based on its frequency of occurrence, the excessive load increase event is considered a Condition II event as defined by the ANS. Even though DNB is the primary concern in the analysis of the excessive load increase, the following three items summarize the criteria associated with this transient:

- The critical heat flux should not be exceeded. This is met by demonstrating that the minimum DNBR does not go below the limit value at any time during the transient.
- Pressure in the RCS and MSS should be maintained below 110 percent of the design pressures.
- The peak linear heat generation rate should not exceed a value that would cause fuel centerline melt.

6.3.10.5 Results

An excessive load increase accident typically did not result in reactor trip, and the plant soon reached a new equilibrium condition at a higher power level based on the increased steam load. Transients assuming manual rod control yield decreased coolant temperatures and pressures resulting from increased heat removal.

A comparison of the plant conditions assuming conservatively bounding deviations in core power, average coolant temperature, and pressure to the conditions corresponding to operation at the safety analysis DNB limit indicated that the minimum DNBR remained above the limit value for each of the cases.

6.3.10.6 Conclusions

It has been demonstrated that for an excessive load increase, the minimum DNBR during the transient will not go below the safety analysis limit value and thus will neither affect fuel cladding integrity nor result in the release of fission products to the RCS.

6.3.11 Rupture of a Steam Pipe

6.3.11.1 Introduction

A steam pipe rupture is assumed to include any accident that results in an uncontrolled steam release from a steam generator. The release can occur as a result of a break in a pipe line or a

valve malfunction. The steam release results in an initial increase in steam flow that decreases during the accident as the steam pressure falls. The removal of energy from the RCS causes a reduction of coolant temperature and pressure. With a negative MTC, the cooldown results in a reduction of core shutdown margin. If the most reactive control rod is assumed to be stuck in its fully withdrawn position, there is a possibility that the core can become critical and return to power even with the remaining control rods inserted. A return to power following a steam pipe rupture is a potential problem only because of the high hot-channel factors that can exist when the most reactive rod is assumed stuck in its fully withdrawn position. Even if the most pessimistic combination of circumstances that could lead to power generation following a steamline break was assumed, the core is ultimately shut down by the boric acid in the SIS.

The analysis of a steam pipe rupture was made to show that assuming the most reactive RCCA stuck in its fully withdrawn position and assuming the worst single failure in the engineered safety features (ESFs), the core cooling capability could be maintained and that offsite doses would not exceed applicable limits. In addition, the analysis considers conditions both with and without offsite power available.

Although DNB and possible clad perforation following a steam pipe rupture are not necessarily unacceptable, the following analysis showed that DNB did not occur, thus ensuring clad integrity.

The following systems provide the necessary protection against a steam pipe rupture:

- SIS actuation from any one of the following:
 - Two-out-of-3 channels of low pressurizer pressure signals
 - Two-out-of-3 high differential pressure signals between steamlines
 - High steam flow in 2-out-of-4 lines (1-out-of-2 per line) in coincidence with either low RCS average temperature (2-out-of-4) or low steamline pressure (2-out-of-4)
 - Two-out-of-3 high containment pressure signals
 - Manual actuation
- The overpower reactor trips (nuclear flux and ΔT) and the reactor trip occurring upon actuation of the SIS.

- Redundant isolation of the main feedwater lines. Sustained high feedwater flow would cause additional cooldown. However, in addition to the normal control action that will close the main feedwater valves, any safety injection signal will rapidly close all feedwater control valves and close the feedwater pump discharge valves, which in turn would trip the main feedwater pumps.
- Closing the fast-acting steamline stop valves (designed to close in less than 5 seconds) on:
 - High steam flow in any 2 steamlines (1-out-of-2 per line) in coincidence with either low RCS average temperature (2-out-of-4) or low steamline pressure (2-out-of-4)
 - Two sets of 2-out-of-3 high-high containment pressure signals.

Each main steamline has a fast-closing stop valve and a check valve. These 8 valves prevent blowdown of more than 1 steam generator for any MSLB location even if 1 valve fails to close. For example, for a MSLB upstream of the stop valve in 1 line, a closure of either the check valve in that line or the stop valves in the other lines will prevent blowdown of the other steam generators.

For breaks downstream of the isolation valves, closure of all valves will completely terminate the blowdown. For any break, in any location, no more than 1 steam generator would experience an uncontrolled blowdown even if one of the isolation valves fails to close.

The effective throat area of the steam generator flow restrictor nozzles is 1.4 ft². These flow areas are considerably less than the main steam pipe area. Thus, the flow restrictor nozzles serve to limit the maximum steam flow for a break at any location.

6.3.11.2 Input Parameters and Assumptions

The following conditions are assumed to exist at the time of a MSLB accident.

EOL shutdown margin at no-load, equilibrium xenon conditions, and the most reactive RCCA stuck in its fully withdrawn position are all assumed. Operation of the control rod banks during core burnup is restricted in such a way that addition of positive reactivity in a steamline break accident will not lead to a more adverse condition than the case analyzed.

The negative moderator coefficient corresponds to an EOL rodded core with the most-reactive RCCA withdrawn. The variation of the coefficient with temperature and pressure is included. The core properties associated with the sector nearest the affected steam generator and those

associated with the remaining sector are conservatively combined to obtain average core properties for reactivity feedback calculations. Furthermore, it is conservatively assumed that the core power distribution was uniform. These two conditions cause an underprediction of the reactivity feedback in the high power region near the stuck rod. To verify the conservatism of this method, the reactivity and power distribution is checked for the limiting statepoints for the cases analyzed.

This core analysis considers the Doppler reactivity from the high fuel temperature near the stuck RCCA, moderator feedback from the high water enthalpy near the stuck RCCA, power redistribution, and non-uniform core inlet temperature effects. For cases in which steam generation occurred in the high flux regions of the core, the effect of void formation is also included. It was determined that the reactivity used in the kinetics analysis was always larger than the reactivity calculated, including the above local effects for the statepoints. These results verified conservatism, that is, an underprediction of negative reactivity feedback from power generation.

Minimum capability for injection of high-concentration boric acid (2400 ppm) solution corresponding to the most restrictive single failure in the High-Head Safety Injection System (HHSIS) is assumed. The Emergency Core Cooling System (ECCS) consists of three systems: the passive accumulators, the Residual Heat Removal System (RHRS), and the High-Head Safety Injection System. Only the HHSIS is modeled for the steamline break accident analysis.

The actual modeling of the HHSIS in RETRAN is described in WCAP-14882-P-A (Reference 1). The flow corresponds to that delivered by 2 degraded HHSI pumps delivering flow to the cold leg header. No credit is taken for the low-concentration borated water, which must be swept from the lines downstream of the RWST prior to the delivery of concentrated boric acid to the RCLs.

For the case in which offsite power is assumed, the sequence of events in the HHSIS is the following. After the generation of the SI signal (appropriate delays for instrumentation, logic, and signal transport included), the appropriate valves began to operate and the HHSI pumps started. In 12 seconds, the valves are assumed to be in the final position and the pump is assumed to be at full speed. In cases where offsite power is not available, an additional 7-second delay is assumed to start the diesels and load the necessary SI equipment onto them.

Design value of the steam generator heat transfer coefficient including allowance for fouling factor is assumed.

Since the steam generators have integral flow restrictors with a 1.4 ft² throat area, any rupture with a break area greater than the area of the flow restrictor, regardless of the location, would

have the same effect on the NSSS as the break equal to the area of the flow restrictor. The following cases were considered in determining the core power and RCS transients.

Case 1: Complete severance of a pipe, with the plant initially at no-load conditions, and full reactor coolant flow with offsite power available.

Case 2: Case 1 with LOOP coincident with the steamline break. LOOP results in RCP coastdown, which was assumed to begin at 3 seconds.

Power peaking factors corresponding to 1 stuck RCCA and non-uniform core inlet coolant temperatures are determined at EOL. The coldest core inlet temperatures are assumed to occur in the sector with the stuck rod. The power peaking factors account for the effect of the local void in the region of the stuck control assembly during the return-to-power phase following the steamline break. This void, in conjunction with the large negative moderator coefficient, partially offsets the effect of the stuck assembly. The power peaking factors depend on the core conditions for power, temperature, pressure, and flow, and thus are different for each case studied.

The core conditions used for both with and without offsite power cases correspond to values determined from the respective transient analyses.

Both cases assumed initial hot shutdown conditions at event initiation since this represents the most conservative initial condition. These hot shutdown initial conditions are considered for cases assuming full-power operation at HFP high T_{avg} of 572°F. Should the reactor be just critical or operating at power at the time of a steamline break, the reactor would be tripped by the normal Overpower Protection System when the power level reaches a trip setpoint. Following a trip at power, the RCS contains more stored energy than at no-load, the average coolant temperature is higher than at no-load, and there is appreciable energy stored in the fuel. Thus, the additional stored energy is removed via the cooldown caused by the steamline break before the no-load conditions of RCS temperature and shutdown margin assumed in the analyses are reached. After the additional stored energy is removed, the cooldown and reactivity insertions proceeded in the same manner as in the analysis, which assumes no-load conditions at time zero. In addition, since the initial steam generator water inventory is greatest at no-load, the magnitude and duration of the RCS cooldown are less for steamline breaks occurring at power.

Perfect moisture separation in the steam generator is assumed.

6.3.11.3 Description of Analysis

The double-ended rupture of a major steamline is the most-limiting cooldown transient. It was analyzed at zero power with no decay heat since decay heat would retard the cooldown, thereby reducing the return to power.

The analysis of the steam pipe break was performed to determine:

- The core heat flux and RCS temperature and pressure resulting from the cooldown following the steamline break. The RETRAN code (Reference 1) was used to calculate the transient conditions.
- The thermal-hydraulic behavior of the core following a steamline break. A detailed thermal-hydraulic digital computer code, VIPRE, was used to determine if DNBR fell below the safety analysis limit for the core conditions computed in the above bulleted paragraph.

6.3.11.4 Acceptance Criteria

A major steamline break is classified as an ANS Condition IV event. Effects of minor secondary system pipe breaks, which are classified as Condition III events, are bounded by the analysis presented in this section.

Conservatively assuming a stuck RCCA with or without offsite power, and assuming a single failure in the ESFs, the core remained in place and intact. Although DNB and possible clad perforation following a steam pipe break are not necessarily unacceptable, the following analysis in fact shows that the DNBR never falls below the safety analysis limit for any break assuming the most reactive assembly stuck in its fully withdrawn position. By meeting the DNB design basis criterion, this analysis condition conservatively met the radiological dose criteria set forth for a steamline break.

6.3.11.5 Results

The calculated sequences of events for both cases are shown in Table 6.3-10.

The results presented were a conservative indication of the events that would occur assuming a steamline break, since it is postulated that all of the conditions described above occur simultaneously.

Core Power and RCS Transient

Figure 6.3-52 shows the core heat flux and core reactivity following a MSLB (complete severance of a steam pipe) at initial no-load conditions. Figure 6.3-53 shows the corresponding vessel inlet temperature and pressurizer pressure after the break occurs. Figure 6.3-54 shows steam flow and steam generator mass of the faulted and intact steam generators during the event. Offsite power was assumed available so that full reactor coolant flow existed. The transient shown assumed an uncontrolled steam release from only 1 steam generator. Should the core be critical at near zero power when the break occurs, the initiation of SI by low steamline pressure will trip the reactor. Steam release from more than 1 steam generator will be prevented by automatic closure of the fast-acting isolation valves in the steamlines by low steamline pressure signals, high containment pressure signals, or high negative steamline pressure rate signals. Even with the failure of 1 valve, release is limited to no more than 10 seconds for the other steam generators while the 1 generator blows down. The steamline stop valves are designed to be fully closed in less than 5 seconds from receipt of a closure signal.

The core attained criticality with the RCCAs inserted (with the design shutdown assuming 1 stuck RCCA) before boron solution at 2400 ppm entered the RCS. A peak core power lower than the nominal full-power value was attained.

The calculation assumed the boric acid was mixed with, and diluted by, the water flowing in the RCS prior to entering the reactor core. The concentration after mixing depended upon the relative flow rates in the RCS and in the HHSIS. The variation of mass flow rate in the RCS due to water density changes was included in the calculation as is the variation of flow rate in the HHSIS due to changes in the RCS pressure. The HHSIS flow calculation included the line losses in the system as well as the pump head curve. Figure 6.3-55 illustrates the core averaged boron concentration during the event.

For the case assuming coincidental LOOP when the SI signal is generated, the SI system delay time included 7 seconds to start the diesel, in addition to the 12 seconds to start the SI pump and open the valves. Criticality was achieved later, and the core power increase was slower, than in the case with offsite power available. The ability of the emptying steam generator to extract heat from the RCS was reduced by the decreased flow in the RCS. The peak power remained well below the nominal full-power value.

It should be noted that following a steamline break, only 1 steam generator blows down completely. Thus, the remaining steam generators were still available for dissipation of decay heat after the initial transient was over. In the case with LOOP, this heat was removed to the atmosphere via the steamline safety valves.

Margin to Critical Heat Flux

DNB analyses were performed for the most conservative of the two analyzed cases, that is, the case with offsite power. The minimum DNBR was greater than the limit value.

6.3.11.6 Conclusions

The analysis showed that the acceptance criteria stated earlier in subsection 6.3.11.4 were satisfied. Although DNB and possible cladding perforation following a steam pipe break were not necessarily unacceptable and not precluded by the criteria, the above analysis showed that the DNBR never fell below the safety analysis limit.

6.3.12 Partial Loss of Reactor Coolant Flow

6.3.12.1 Introduction

A partial loss-of-forced-reactor-coolant-flow accident can result from a mechanical or electrical failure in an RCP, or from a fault in the power supply to these pumps. If the reactor is at power at the time of the event, the immediate effect is a rapid increase in coolant temperature. This increase in coolant temperature could result in DNB, with subsequent fuel damage, if the reactor is not promptly tripped.

The reactor trip on low reactor coolant flow provides protection against partial loss-of-flow conditions. This function is generated by 2-out-of-3 low-flow signals per RCL. Above Permissive P-8, low flow in any loop will actuate a reactor trip. Between approximately 10 percent power (Permissive P-7) and the power level corresponding to Permissive P-8, low flow in any 2 loops will actuate a reactor trip.

6.3.12.2 Input Parameters and Assumptions

This accident is analyzed using the RTDP (Reference 3). Initial core power (consistent with SPU conditions) and reactor coolant pressure are assumed to be at their nominal values for steady-state, full-power operation. Reactor coolant temperature is assumed to be at the nominal value for the high T_{avg} program. Uncertainties in initial conditions are included in the DNBR limit as described in the RTDP (Reference 3). MMF is also assumed. A conservatively large absolute value of the Doppler-only power coefficient is used along with the most positive MTC limit for full-power operation (0 pcm/°F). These assumptions maximize the core power during the initial part of the transient when the minimum DNBR is reached.

A conservatively low trip reactivity value (4.0-percent $\Delta\rho$) is used to minimize the effect of rod insertion following reactor trip and maximize the heat flux statepoint used in the DNBR evaluation for this event. This value is based on the assumption that the highest worth RCCA is stuck in its fully withdrawn position. A conservative trip reactivity worth versus rod position is modeled in addition to a conservative rod drop time (2.4 seconds to dashpot). The trip reactivity versus rod position curve is confirmed to be valid as part of the RSAC verification process.

In addition, a 0.85-percent core flow penalty is modeled to account for RCS loop-to-loop flow asymmetry.

Normal reactor control systems and engineered safety systems (for example, SI) are not required to function. No single active failure in any system or component required for mitigation will adversely affect the consequences of this event.

6.3.12.3 Description of Analysis

A partial loss-of-flow involving the loss of 1 RCP with 4 loops in operation was analyzed for the SPU conditions.

The transient was analyzed using 2 computer codes. First, the RETRAN computer code (Reference 1) was used to calculate the loop and core flow transients, nuclear power transient, and primary system pressure and temperature transients. The VIPRE computer code (Reference 8) was then used to calculate the hot channel heat flux transient and DNBR, based on the nuclear power and RCS flow from RETRAN. The DNBR transient presented was based on the minimum of the typical and thimble cells.

6.3.12.4 Acceptance Criteria

A partial loss-of-forced-reactor-coolant-flow incident is classified by the ANS as a Condition II event. The immediate effect is a rapid increase in reactor coolant temperature and subsequent increase in RCS pressure. The primary acceptance criterion for this event is that the critical heat flux should not be exceeded. This was ensured by demonstrating that the minimum DNBR did not go below the applicable safety analysis limit at any time during the transient. The analysis results also demonstrated that pressure in the RCS and MSS remained below 110 percent of the respective design pressures to ensure that the applicable Condition II pressure criteria were met.

6.3.12.5 Results

The partial loss-of-forced-reactor-coolant-flow event was the least DNB-limiting transient among all loss-of-flow cases. Reactor trip for the partial loss-of-flow case occurred on a low primary coolant flow signal. The VIPRE analysis confirmed that the minimum DNBR was greater than the safety analysis limit. Fuel clad damage criteria were not challenged in the partial loss-of-forced-reactor-coolant-flow event since the DNB criterion was met.

The analysis of the partial loss-of-flow event also demonstrated that the peak RCS and MSS pressures were well below their respective limits.

The sequence of events for the partial loss-of-flow transient is presented in Table 6.3-11. The transient results for this case are presented in Figures 6.3-56 through 6.3-58.

6.3.12.6 Conclusions

The analysis performed at SPU conditions demonstrated that, for the partial loss-of-flow incident, the DNBR did not decrease below the safety analysis limit at any time during the transient; thus, no fuel or clad damage is predicted. The peak primary and secondary system pressures remained below their respective limits at all times. All applicable acceptance criteria were therefore met.

6.3.13 Complete Loss-of-Reactor-Coolant Flow

6.3.13.1 Introduction

A complete loss-of-forced-reactor-coolant-flow accident can result from simultaneous loss of electrical power or a reduction in supply frequency to all RCPs. If the reactor is at power at the time of the event, the immediate effect is a rapid increase in coolant temperature. This increase in coolant temperature could result in DNB, with subsequent fuel damage, if the reactor is not promptly tripped.

The following signals provide protection against a complete loss-of-forced-reactor-coolant-flow incident:

- Low voltage on pump power supply bus (above Permissive P-7)
- Low reactor coolant flow (1-out-of-4 above Permissive P-8, 2-out-of-4 above Permissive P-7).

- RCP circuit breakers opening (1-out-of-4 above Permissive P-8, 2-out-of-4 above Permissive P-7).

The reactor trip on RCP undervoltage protects against conditions that can cause a loss of voltage to all RCPs, that is, LOOP. The reactor trip on RCP underfrequency is provided to protect against frequency disturbances on the power grid.

The reactor trip on low primary coolant loop flow provides protection against loss-of-flow conditions that affect individual RCLs and serves as a backup for the undervoltage and underfrequency trip functions. The reactor trip on low primary coolant loop flow is generated by 2-out-of-3 low-flow signals per RCL. Above Permissive P-8, low flow in any loop will actuate a reactor trip. Between approximately 10-percent power (Permissive P-7) and the power level corresponding to Permissive P-8, low flow in any 2 loops will actuate a reactor trip.

6.3.13.2 Input Parameters and Assumptions

This accident is analyzed using the RTDP (Reference 3). Initial core power (consistent with uprated power conditions) and reactor coolant pressure are assumed to be at their nominal values for steady-state, full-power operation. Reactor coolant temperature is assumed to be at the nominal value for the high T_{avg} program. Uncertainties in initial conditions are included in the DNBR limit as described in the RTDP (Reference 3). MMF is also assumed. A conservatively large absolute value of the Doppler-only power coefficient is used along with the most positive (MTC) limit for full-power operation (0 pcm/°F). These assumptions maximize the core power during the initial part of the transient when the minimum DNBR was reached.

A conservatively low trip reactivity value (4.0-percent $\Delta\rho$) is used to minimize the effect of rod insertion following reactor trip and maximize the heat flux statepoint used in the DNBR evaluation for this event. This value is based on the assumption that the highest worth RCCA is stuck in its fully withdrawn position. A conservative trip reactivity worth versus rod position is modeled in addition to a conservative rod drop time (2.4 seconds to dashpot). The trip reactivity versus rod position curve is confirmed to be valid as part of the RSAC verification process.

Normal RCS and engineered safety systems (for example, SI) are not required to function. No single active failure in any system or component required for mitigation will adversely affect the consequences of this event.

6.3.13.3 Description of Analysis

The following complete loss-of-forced-reactor-coolant-flow cases were analyzed for the SPU.

- Complete loss-of-flow transient due to a complete loss of power to all RCPs with 4 loops in operation
- Complete loss-of-flow transient due to an underfrequency condition

Case 1 assumed that the RCPs begin to coast down upon reaching an undervoltage trip setpoint (modeled to occur at $t = 0$ seconds in this analysis). Rod motion following the undervoltage trip was modeled to occur at $t = 1.5$ seconds, reflecting an undervoltage trip time delay of 1.5 seconds. For the underfrequency event (Case 2), a frequency decay rate of 5 Hz/sec was assumed to begin at $t = 0$ seconds, decreasing pump speed, and thus, flow to all loops. At $t = 0.6$ seconds, the underfrequency trip setpoint of 57.0 Hz was reached. Rod motion occurred at $t = 1.6$ seconds, following a 1-second underfrequency trip time delay.

The transients were analyzed using 2 computer codes. First, the RETRAN computer code (Reference 1) was used to calculate the loop and core flow transients, nuclear power transient, and primary system pressure and temperature transients. The VIPRE computer code (Reference 8) was then used to calculate the hot-channel-heat-flux-transient and DNBR, based on the nuclear power and RCS flow from RETRAN. The DNBR transient was based on the minimum of the typical and thimble cells.

6.3.13.4 Acceptance Criteria

A complete-loss-of-forced-reactor-coolant-flow incident is classified by the ANS as a Condition III event. However, since a Condition II LOOP event could lead to a Condition III complete-loss-of-flow-event, the incident is analyzed to meet the more restrictive Condition II criteria to bound the complete loss of flow following a LOOP event.

The immediate effect from a complete-loss-of-forced-reactor-coolant flow is a rapid increase in reactor coolant temperature and subsequent increase in RCS pressure. The primary acceptance criterion for this event is that the critical heat flux should not be exceeded. This was ensured by demonstrating that the minimum DNBR did not go below the applicable safety analysis limit at any time during the transient.

The analysis results also demonstrated that pressure in the RCS and MSS remained below 110 percent of the respective design pressures to ensure that the applicable Condition II pressure criteria were met.

6.3.13.5 Results

For the IP2 SPU, both the undervoltage and frequency decay transients were analyzed. The VIPRE analyses for these scenarios confirmed that the minimum DNBR values were greater than the safety analysis limit.

The analysis of the complete-loss-of-flow event also demonstrated that the peak RCS and MSS pressures were well below their respective limits.

The sequence of events for the more limiting complete-loss-of-flow case, the frequency decay transient, is presented in Table 6.3-12. The transient results for this case are presented in Figures 6.3-59 through 6.3-61.

6.3.13.6 Conclusions

The analysis of the undervoltage and frequency decay cases, performed at SPU conditions, demonstrated that the DNBR did not decrease below the safety analysis limit at any time during the transients, thus, the integrity of the fuel was maintained. The peak primary and secondary system pressures remained below their respective limits at all times. Therefore, all applicable acceptance criteria were met.

6.3.14 Locked Rotor Accident

6.3.14.1 Introduction

The event postulated is an instantaneous seizure of a RCP rotor or the sudden break of a RCP shaft. Flow through the affected RCL is rapidly reduced, leading to initiation of a reactor trip on a low RCL flow signal.

Following initiation of the reactor trip, heat stored in the fuel rods continues to be transferred to the coolant causing the coolant to expand. At the same time, heat transfer to the shell-side of the steam generators is reduced; first because the reduced primary flow results in a decreased tube-side film coefficient, and secondly because the reactor coolant in the tubes cools down while the shell-side temperature increases (turbine steam flow is reduced to zero upon plant trip due to turbine trip on reactor trip). The rapid expansion of coolant in the reactor core, combined with reduced heat transfer in the steam generators, causes an insurge into the pressurizer and a pressure increase throughout the RCS. The insurge into the pressurizer compresses the steam volume, actuates the automatic spray system, opens the PORVs, and opens the PSVs, in that sequence. The 2 PORVs are designed for reliable operation and would be expected to function properly during the event. However, for conservatism, their pressure-reducing effect,

as well as the pressure-reducing effect of the pressurizer spray, was not included in the analysis.

The consequences of a locked rotor (that is, an instantaneous seizure of a pump shaft) are very similar to those of a pump shaft break. The initial rate of reduction in coolant flow is slightly greater for the locked rotor event. However, with a broken shaft, the impeller could conceivably be free to spin in the reverse direction. The effect of reverse spinning is to decrease the steady-state core flow when compared to the locked rotor scenario. The analysis considered the most-limiting combination of conditions for the locked rotor and pump-shaft break events.

6.3.14.2 Input Parameters and Assumptions

Two cases are evaluated in the analysis. Both assumed 1 locked RCP rotor/shaft break with a total of 4 loops in operation.

The first case is analyzed to evaluate the RCS pressure and fuel clad temperature transient conditions. This case is analyzed using the STDP. Initial core power, reactor coolant temperature, and pressure are assumed to be at their maximum values consistent with the uprated full-power conditions including allowances for calibration and instrument errors. This assumption results in a conservative calculation of fuel clad temperature transient conditions and of the coolant surge into the pressurizer, which in turn results in a maximum calculated peak RCS pressure. In addition, a 0.85-percent core flow penalty is modeled to account for RCS loop-to-loop flow asymmetry.

The second case is an evaluation of DNB in the core during the transient. This case is analyzed using the RTDP (Reference 3). Initial core power (consistent with SPU conditions) and reactor coolant pressure are assumed to be at their nominal values for steady state, full-power operation. Reactor coolant temperature is assumed to be at the nominal value for the high T_{avg} program. Uncertainties in initial conditions are included in the DNBR limit as described in the RTDP (Reference 3). A conservatively large absolute value of the Doppler-only power coefficient is used along with the most-positive MTC limit for full-power operation (0 pcm/°F). These assumptions maximized the core power during the initial part of the transient when the minimum DNBR is reached. In addition, a 0.85-percent core flow penalty is modeled to account for RCS loop-to-loop flow asymmetry.

A conservatively low-trip reactivity value (4.0-percent $\Delta\rho$) is used to minimize the effect of rod insertion following reactor trip and maximize the heat flux statepoint used in the DNBR evaluation for this event. This value is based on the assumption that the highest worth RCCA is stuck in its fully withdrawn position. A conservative trip reactivity worth versus rod position is

modeled in addition to a conservative rod drop time (2.4 seconds to dashpot). The trip reactivity versus rod position curve is confirmed to be valid as part of the RSAC verification process.

Normal reactor control systems and engineered safety systems (for example, SI) are not required to function. No single active failure in any system or component required for mitigation will adversely affect the consequences of this event.

6.3.14.3 Description of Analysis

The transients were analyzed using 2 computer codes. First, the RETRAN computer code (Reference 1) was used to calculate the loop and core flow transients, nuclear power transient, and primary system pressure and temperature transients. The VIPRE computer code (Reference 8) was then used to calculate the hot channel heat flux transient and DNBR, based on the nuclear power and RCS flow from RETRAN. The DNBR transient is based on the minimum DNBR of the typical and thimble cells.

For the peak RCS pressure evaluation, the initial pressure was conservatively estimated as 28 psi above the nominal pressure of 2250 psia to allow for errors in pressurizer pressure measurement and control channels. This provides the highest possible rise in the coolant pressure during the transient. The pressure response reported in Table 6.3-13 was for the point in the RCS having the maximum pressure, that is, the outlet of the RCP.

For a conservative analysis of fuel rod behavior, the hot spot evaluation assumed that DNB occurred at initiation of the transient and continues throughout the event. This assumption reduces heat transfer to the coolant and results in conservatively high hot spot temperatures.

Evaluation of the Pressure Transient

After pump seizure, coolant flow in the loop with the faulted RCP decreased rapidly and RCS temperature and pressure increased. A reactor trip signal was generated when the flow in the affected loop reached 85 percent of nominal flow. Rod motion began 1 second later and the neutron flux was rapidly reduced by control rod insertion. As RCS pressure increased, no credit was taken for the pressure-reducing effect of pressurizer PORVs or pressurizer spray, nor was credit taken for steam dump or controlled feedwater flow after plant trip. Although these systems are expected to function and would result in a lower peak pressure, an additional degree of conservatism was provided by not including their effect.

Evaluation of DNB in the Core during the Event

For this event, DNB was assumed to occur in the core; therefore, an evaluation of the consequences with respect to fuel rod thermal transients was performed. Results obtained from analysis of this hot spot condition represent the upper limit with respect to clad temperature and zirconium-water reaction. In the evaluation, the rod power at the hot spot conservatively considers an F_Q of 2.50. The number of rods-in-DNB is conservatively calculated for use in dose consequence evaluations.

6.3.14.4 Acceptance Criteria

The RCP locked rotor accident is classified by the ANS as a Condition IV event. The following items summarize the criteria associated with this event:

- Fuel cladding damage, including melting, due to increased reactor coolant temperatures must be prevented. This is precluded by demonstrating that the maximum clad temperature at the core hot spot remains below 2700°F, and the zirconium-water reaction at the core hot spot is less than 16 weight percent.
- Pressure in the RCS should be maintained below that which would cause stresses to exceed the faulted condition stress limits.
- Rods-in-DNB should be less than or equal to that assumed in the radiological dose analyses for the locked rotor/shaft break event.

6.3.14.5 Results

The results of the locked rotor/shaft break analysis are summarized in Table 6.3-13 and demonstrate that the acceptance criteria documented in subsection 6.3.14.4 continued to be met for the SPU. The number of rods-in-DNB (calculated as 0-percent rods-in-DNB) was less than that supported by the radiological dose analysis. Hence, the rods-in-DNB criterion was also met for the locked rotor/shaft break event. The calculated sequence of events is presented in Table 6.3-14 for the locked rotor event. The transient results for the peak-pressure/hot-spot case are provided in Figures 6.3-62 through 6.3-64.

6.3.14.6 Conclusions

The analysis performed at SPU conditions demonstrated that, for the locked rotor/shaft break event, the peak clad temperature calculated for the hot spot during the worst transient remained

considerably less than 2700°F and the amount of zirconium-water reaction was small. Under such conditions, the core will remain in place and intact with no loss-of-core-cooling capability.

The analysis also confirmed that the peak RCS pressure reached during the transient was less than that which would cause stresses to exceed the faulted condition stress limits. The rods-in-DNB design criterion was also met.

The protection features previously described in subsection 6.3.14.1 provided mitigation for a locked rotor/shaft break transient such that the above criteria were satisfied.

6.3.15 Rupture of a CRDM Housing – RCCA Ejection

6.3.15.1 Introduction

This accident is defined as a mechanical failure of a CRDM pressure housing resulting in the ejection of the RCCA and drive shaft. The consequence of this mechanical failure is a rapid positive reactivity insertion together with an adverse core power distribution, possibly leading to localized fuel rod damage. The resultant core thermal power excursion is limited by the Doppler reactivity effect of the increased fuel temperature, and terminated by reactor trip actuated by high nuclear power signals.

A failure of a CRDM housing sufficient to allow a control rod to be rapidly ejected from the core is not considered credible for the following reasons.

- Each full-length mechanism housing is completely assembled and shop-tested at 4100 psig.
- The mechanism housings are individually hydrotested after they are attached to the head adapters in the reactor vessel head and checked during the hydrotest of the completed RCS.
- Stress levels in the mechanism are not affected by anticipated system transients at power or by thermal movement of the coolant loops. Moments induced by the design earthquake can be accepted within the allowable primary working stress ranges specified in the ASME Code, Section III, for Class I components.
- The latch mechanism housing and rod travel housing are each a single length of forged type-304 stainless steel. This material exhibits excellent notch toughness at all temperatures that will be encountered.

A significant margin of strength in the elastic range, together with the large energy absorption capability in the plastic range, gives additional assurance that gross failure of the housing will not occur. The joints between the latch mechanism housing and rod travel housing are threaded joints reinforced by canopy-type rod welds.

In general, the reactor is operated with the RCCAs inserted only far enough to control design neutron flux shape. Reactivity changes caused by core depletion are compensated for by boron changes. Furthermore, the location and grouping of control rod banks are selected during the nuclear design to lessen the severity of a RCCA ejection accident. Therefore, should a RCCA be ejected from its normal position during full-power operation, only a minor reactivity excursion, at worst, could be expected to occur. The position of all RCCAs is continuously indicated in the control room. An alarm will occur if a bank of RCCAs approaches its insertion limit or if one 1 control rod assembly deviates from its bank. There are low and low-low level insertion alarm circuits for each bank. The control rod position monitoring and alarm systems are described in WCAP-7588 (Reference 10).

6.3.15.2 Input Parameters and Assumptions

Input parameters for the analysis are conservatively selected on the basis of values calculated for this type of core. The most important parameters are discussed below. Table 6.3-15 lists the parameters used in this analysis.

Ejected Rod Worths and Hot Channel Factors

The values for ejected rod worths and hot channel factors are calculated using either 3-D static methods or a synthesis of 1-D and 2-D calculations. Standard nuclear design codes are used in the analysis. No credit is taken for the flux-flattening effects of reactivity feedback. The calculation is performed for the maximum allowed bank insertion at a given power level, as determined by the rod insertion limits. The analysis assumes adverse xenon distributions to provide worst-case results.

Delayed Neutron Fraction, β

The ejected rod accident is sensitive to β if the ejected rod worth is equal to or greater than β , as in the zero-power transients. To allow for future fuel cycle flexibility, conservative estimates of β of 0.50 percent at beginning of cycle, 0.40 percent at HFP and 0.42 percent at end of cycle HZP are used in the analysis.

Reactivity Weighting Factor

The largest temperature rise, and hence the largest reactivity feedbacks occur in channels where the power is higher than average. Since the weight of a region is dependent on flux, these regions have high weights. This means that the reactivity feedback is larger than that indicated by a simple single-channel analysis. Physics calculations have been performed for temperature changes with a flat temperature distribution and with a large number of axial and radial temperature distributions. Reactivity changes are compared and effective weighting factors determined. These weighting factors take the form of multipliers that, when applied to single-channel feedbacks, accounted for the effective whole-core feedbacks for the appropriate flux shape.

In this analysis, a 1-D (axial) spatial kinetics method is employed; thus axial weighting is not necessary if the initial condition is made to match the ejected rod configuration. In addition, no weighting is applied to moderator feedback. A conservative radial weighting factor is applied to the transient fuel temperature to obtain an effective fuel temperature, as a function of time, accounting for the missing spatial dimension. These weighting factors have also been shown to be conservative when compared to 3-D analysis (Reference 10).

Moderator and Doppler Coefficient

The critical boron concentrations at the BOL and EOL are adjusted in the nuclear code to obtain moderator density coefficient curves that are conservative when compared to the actual design conditions for the plant. As discussed above, no weighting factor was applied to these results.

The Doppler reactivity defect is determined as a function of power level using a 1-D steady-state computer code with a Doppler weighting factor of 1.0. The Doppler weighting factor will increase under accident conditions, as discussed above.

Heat Transfer Data

The FACTRAN (Reference 5) code used to determine the hot spot transient contains standard curves of thermal conductivity versus fuel temperature. During a transient, the peak centerline fuel temperature is independent of gap conductance during the transient. The cladding temperature is, however, strongly dependent on gap conductance and is highest for high-gap conductance. For conservatism, a high-gap heat transfer coefficient of 10,000 Btu/hr-ft²-°F has been used during transients. This value corresponds to a negligible gap resistance and a further increase would have essentially no effect on the rate of heat transfer.

Coolant Mass Flow Rates

When the core is operating at full power, all 4 coolant pumps will always be operating. For zero-power conditions, the system is conservatively assumed to be operating with 2 pumps. The principal effect of operating at reduced flow is to reduce the film-boiling heat transfer coefficient. This results in higher peak cladding temperatures, but does not affect peak centerline fuel temperature. Reduced flow also lowered the critical heat flux. However, since DNB is always assumed at the hot spot and the heat flux rises very rapidly during the transient, this produces only second-order changes in the cladding and centerline fuel temperatures. All zero-power analyses for both average core and the hot spot are conducted assuming 2 pumps in operation.

Trip Reactivity Insertion

The trip reactivity insertion is assumed to be 4 percent $\Delta K/K$ from HFP and 2 percent $\Delta K/K$ from HZP, including the effect of 1 stuck RCCA. These values are also reduced by the ejected rod. The shutdown reactivity is simulated by dropping a rod of the required worth into the core. The start of rod motion occurs 0.5 seconds after reaching the power-range high-neutron-flux trip setpoint. It is assumed that insertion to dashpot occurs 2.4 seconds after the rods began to fall. The time delay to full insertion, combined with the 0.5-second trip delay, conservatively delays insertion of shutdown reactivity into the core.

The minimum design shutdown margin available for this plant at HZP may only occur at EOL in the equilibrium cycle. This value includes an allowance for the worst stuck rod, an adverse xenon distribution, conservative Doppler and moderator defects, and an allowance for calculational uncertainties. Physics calculations have shown that 2 stuck RCCAs (1 of which is the worst ejected rod) reduce the shutdown margin by about an additional 1 percent $\Delta K/K$. Therefore, following a reactor trip resulting from an RCCA ejection accident, the reactor will be subcritical when the core returns to HZP.

6.3.15.3 Description of Analysis

This section describes the models used in the analysis of the rod ejection accident. Only the initial few seconds of the power transient are discussed, since the long-term considerations are the same as for a LOCA.

The calculation of the RCCA ejection transient was performed in 2 stages, first an average core channel calculation and then a hot region (hot spot) calculation. The average core calculation used spatial neutron-kinetics methods to determine average power generation versus time including the various total core feedback effects, that is, Doppler reactivity and moderator reactivity. Enthalpy and temperature transients at the hot spot were then determined by

multiplying the average core energy generation by the hot channel factor and by performing a fuel rod transient heat transfer calculation. The power distribution calculated without feedback was conservatively assumed to exist throughout the transient. A detailed discussion of the method of analysis can be found in WCAP-7588, Revision 1-A (Reference 10).

Average Core Analysis

The spatial-kinetics computer code, TWINKLE (Reference 4) was used for the average core transient analysis. This code solves the two-group neutron diffusion theory kinetic equation in 1, 2, or 3 spatial dimensions (rectangular coordinates) for 6 delayed neutron groups and up to 8000 spatial points. The computer code includes a detailed multi-region, transient fuel-clad-coolant-heat-transfer model for calculation of point-wise Doppler and moderator feedback effects. This analysis used the code as a 1-D axial kinetics code since it allows a more-realistic representation of the spatial effects of axial moderator feedback and RCCA movement. However, since the radial dimension was missing, it was still necessary to use very conservative methods (described below) for calculating the ejected rod worth and hot channel factor.

Hot Spot Analysis

In the hot spot analysis, the initial heat flux is equal to the nominal heat flux times the design hot channel factor. During the transient, the heat flux hot channel factor is linearly increased to the transient value in 0.1 second, the time for full ejection of the rod. Therefore, the assumption is made that the hot spot conditions before and after ejection are coincident. This is very conservative since the peak nuclear power after ejection will occur in or adjacent to the assembly with the ejected rod, whereas prior to ejection the power in this region will be depressed.

The average core energy addition, calculated as described above, is multiplied by the appropriate hot channel factors. The hot spot analysis uses the detailed fuel and clad transient heat transfer computer code, FACTRAN (Reference 5). This computer code calculates the transient temperature distribution in a cross section of a metal-clad UO_2 fuel rod and the heat flux at the surface of the rod, using as input the nuclear power versus time and local coolant conditions. The zirconium-water reaction is explicitly represented, and all material properties are represented as functions of temperature. A conservative pellet radial power distribution is assumed within the fuel rod.

FACTRAN uses the Dittus-Boelter or Jens-Lottes correlation to determine the film heat transfer before DNB, and the Bishop-Sandberg-Tong correlation (Reference 11) to determine the film-boiling coefficient after DNB. The use of the Bishop-Sandberg-Tong correlation conservatively assumes zero bulk fluid quality. The DNB heat flux was not calculated; instead, the code was

forced into DNB by specifying a conservative DNB heat flux. The gap heat transfer coefficient could be calculated by the code; however, it was adjusted to force the full-power, steady-state temperature distribution to agree with fuel heat transfer design codes.

Reactor Protection

The protection for this accident, as explicitly modeled in the analysis, was provided by the power-range high-neutron-flux trip (high and low settings). This protection function is part of the Reactor Trip System. No single failure of the Reactor Trip System will negate the protection function required for the rod ejection accident, or adversely affect the consequences of the accident.

6.3.15.4 Acceptance Criteria

Due to the extremely low probability of an RCCA ejection accident, this event is classified as an ANS Condition IV event. As such, some fuel damage could be considered an acceptable consequence.

The Idaho Nuclear Corporation (Reference 12) has carried out comprehensive studies of the threshold of fuel failure and the threshold of significant conversion of the fuel thermal energy to mechanical energy as part of the SPERT project. Extensive tests of UO₂ zirconium-clad fuel rods representative of those present in PWR-type cores have demonstrated failure thresholds in the range of 240 to 257 cal/gm. However, other rods of a slightly different design exhibited failure as low as 225 cal/gm. These results differ significantly from the TREAT (Reference 13) results, which indicated a failure threshold of 280 cal/gm. Limited results have indicated that this threshold decreased 10 percent with fuel burnup. The clad failure mechanism appears to be melting for unirradiated (zero burnup) rods and brittle fracture for irradiated rods. The conversion ratio of thermal to mechanical energy is also important. This ratio becomes marginally detectable above 300 cal/gm for unirradiated rods, and 200 cal/gm for irradiated rods; catastrophic failure (large fuel dispersal, large pressure rise), even for irradiated rods, did not occur below 300 cal/gm.

The real physical limits of this accident are that the rod ejection event and any consequential damage to either the core or the RCS must not prevent long-term core cooling and any offsite dose consequences must be within the guidelines of 10CFR100. More specific and restrictive criteria are applied to ensure fuel dispersal in the coolant, gross lattice distortion or severe shock waves will not occur. In view of the above experimental results, and the conclusions of WCAP-7588, Revision 1-A (Reference 10) and Westinghouse letter NS-NRC-84-3466 (Reference 14), the limiting criteria are:

- Average fuel pellet enthalpy at the hot spot must be maintained below 200 cal/gm.
- Peak reactor coolant pressure must be less than that which could cause RCS stresses to exceed the faulted-condition stress limits.
- Fuel melting is limited to less than 10 percent of the fuel volume at the hot spot even if the average fuel pellet enthalpy is below the limits of the criterion in the first bulleted paragraph.

6.3.15.5 Results

A summary of the parameters used in the rod ejection analyses and the analysis results are listed in Table 6.3-15. For both HFP cases, control bank D was assumed at its insertion limit. For both HZP cases, control bank D was assumed fully inserted with banks B and C at their insertion limits.

The nuclear power and hot spot fuel and clad temperature transients for all 4 cases, BOL HFP, BOL HZP, EOL HFP, and EOL HZP are shown in Figures 6.3-65 through 6.3-72. The sequence of events for all 4 cases are listed in Tables 6.3-16 and 6.3-17.

For all 4 cases, peak hot spot average enthalpy was less than the acceptance criteria limit of 200 cal/gm (360 Btu/lb) (maximum). The peak fuel centerline temperature for the HFP cases exceeded the conservative assumed temperature for fuel melt (4900°F at BOL; 4800°F at EOL), but the predicted fuel melt was less than the acceptance criteria limit of 10-percent fuel pellet volume (maximum) at the hot spot. The peak fuel centerline temperature for the HZP cases remained below the conservative assumed temperature for fuel melt (4900°F at BOL; 4800°F at EOL) and resulted in no fuel pellet melt at the hot spot.

A detailed calculation of the pressure surge for an ejected rod worth of 1 dollar at BOL HFP, indicated that the peak pressure did not exceed that which would cause RPV stress to exceed the faulted condition stress limits (Reference 10). Since the severity of the present analysis did not exceed the worst-case analysis, the accident for this plant will not result in an excessive pressure rise or further adverse effects to the RCS.

6.3.15.6 Conclusions

Despite the conservative assumptions, the analyses indicated that the described fuel and clad limits were not exceeded. It was concluded that there is no danger of sudden fuel dispersal into the coolant. Since the peak pressure did not exceed that which would cause stresses to exceed the faulted condition stress limits, it was concluded that there is no danger of further

consequential damage to the RCS. The analyses demonstrated that the fission product release as a result of fuel rods entering DNB was limited to less than 10 percent of the fuel rods in the core.

6.3.16 References

1. WCAP-14882-P-A (Proprietary), *RETRAN-02 Modeling and Qualification for Westinghouse Pressurized Water Reactor Non-LOCA Safety Analyses*, D. S. Huegel, et al., April 1999.
2. WCAP-9272-P-A (Proprietary), *Westinghouse Reload Safety Evaluation Methodology*, F. M. Bordelon et. al., July 1985.
3. WCAP-11397-A (Nonproprietary) and WCAP-11397-P-A (Proprietary), *Revised Thermal Design Procedure*, A. J. Friedland and S. Ray, April 1989.
4. WCAP-7979-P-A (Proprietary) and WCAP-8028-A (Nonproprietary), *TWINKLE, A Multi-Dimensional Neutron Kinetics Computer Code*, R. F. Barry, Jr. and D. H. Risher, January 1975
5. WCAP-7908-A, *FACTRAN - A FORTRAN -IV Code for Thermal Transients in a UO₂ Fuel Rod*, H. G. Hargrove, December 1989.
6. WCAP-7907-P-A (Proprietary) and WCAP-7907-A (Nonproprietary), *LOFTRAN Code Description*, T. W. T. Burnett, et al., April 1984.
7. WCAP-11394 (Proprietary) and WCAP-11395 (Nonproprietary), *Methodology for the Analysis of the Dropped Rod Event*, R. L. Haessler, et al, April 1987.
8. WCAP-14565-A (Proprietary) and WCAP-15306 (Nonproprietary), *Vipre-01 Modeling and Qualification for Pressurized Water Reactor Non-LOCA Thermal-Hydraulic Safety Analysis*, Sung, Y. X., et al., October 1999.
9. ANSI/ANS-5.1 - 1979, *American National Standard for Decay Heat Power in Light Water Reactors*, August 1979.
10. WCAP-7588, *An Evaluation of the Rod Ejection Accident in Westinghouse Pressurized Water Reactors using Special Kinetics Methods*, Rev. 1A, D. H. Risher, January 1975.
11. ASME 65-HT-31, *Forced Convection Heat Transfer at High Pressure After the Critical Heat Flux*, A. A. Bishop, R. O. Sandberg, and L. S. Tong, August 1965.

12. *Annual Report - SPERT Project, October 1968 - September 1969*, IN-1370 Idaho Nuclear Corporation, T. G. Taxebius, ed., June 1970.
13. ANL-7225, P 177, *Studies in TREAT of Zircaloy 2-Clad, UO₂-Core Simulated Fuel Elements*, R. C. Liimatainen and F. J. Testa, November 1966.
14. Letter NS-NRC-89-3466, *Use of 2700°F PCT Acceptance Limit in Non-LOCA Accidents*, from W. J. Johnson of Westinghouse Electric Corporation to Mr. R. C. Jones of the NRC, October 23, 1989.

**Table 6.3-1
List of Non-LOCA Events**

| Licensing Report Section | Event | UFSAR Section | Non-LOCA Computer Code |
|---------------------------------|--|----------------------|-------------------------------|
| 6.3.2 | Uncontrolled RCCA Withdrawal from a Subcritical or Low-Power Startup Condition | 14.1.1 | TWINKLE FACTRAN |
| 6.3.3 | Uncontrolled RCCA Withdrawal at Power | 14.1.2 | RETRAN |
| 6.3.4 | RCCA Drop/Misoperation | 14.1.4 | LOFTRAN |
| 6.3.5 | CVCS Malfunction | 14.1.5 | N/A |
| 6.3.6 | Loss of External Electrical Load | 14.1.8 | RETRAN |
| 6.3.7 | LONF | 14.1.9 | RETRAN |
| 6.3.8 | LOAC to the Station Auxiliaries | 14.1.12 | RETRAN |
| 6.3.9 | Excessive Heat Removal Due to Feedwater System Malfunctions | 14.1.10 | RETRAN |
| 6.3.10 | Excessive Load Increase Incident | 14.1.11 | N/A |
| 6.3.11 | Rupture of a Steam Pipe | 14.2.5 | RETRAN |
| 6.3.12 | Partial Loss of Reactor Coolant Flow | 14.1.6 | RETRAN |
| 6.3.13 | Complete Loss of Reactor Coolant Flow | 14.1.6 | RETRAN |
| 6.3.14 | Locked Rotor Accident | 14.1.6.5 | RETRAN |
| 6.3.15 | Rupture of a Control Rod Mechanism Housing – RCCA Ejection | 14.2.6 | TWINKLE FACTRAN |

Note:

No evaluations were performed for UFSAR Section 14.1.3, "Incorrect Positioning of Part-Length Rods" or 14.1.7, "Startup of an Inactive Reactor Coolant Loop." IP2 has since removed part-length rods from the reactor and, per Technical Specifications, requires that all 4 RCPs be operating for reactor power operation. See UFSAR Sections 14.1.3 and 14.1.7.

Table 6.3-2

Trip Setpoint and Maximum Time Delay for Non-LOCA Safety Analysis

| Reactor Trip Function | Time Delay (seconds) | Maximum Trip Setpoint Assumed for Analysis |
|--|---------------------------------|---|
| Power Range Flux (high setting) | 0.5 | 116% |
| Power Range Flux (low setting) | 0.5 | 35% |
| OTΔT | 10.5 ⁽¹⁾ | Variable (see above) |
| OPΔT | 10.5 ¹ | Variable (see above) |
| High-Pressurizer Pressure | 2.0 | 2440 psia |
| Low-Pressurizer Pressure | 2.0 | 1860 psia |
| Low Reactor Coolant Flow | 1.0 | 85% of loop flow |
| Low-Low Steam Generator Water Level (LONF/LOOP events) | 2.0 | 0% NRS |
| High-High Steam Generator Water Level (feedwater isolation) | 10.0 | 90% NRS |
| (turbine trip) | 2.0 | 90% NRS |
| Reactor Trip (following turbine trip) | 2.0 | N/A |

Note:

1. Delay includes RTD response, filter time constant setting, and electronic delay.

**Table 6.3-3
Non-LOCA Key Accident Analysis Assumptions
for IP2 SPU**

| | |
|--|------------------------------------|
| NSSS Power | 3230 MWt |
| Reactor Power | 3216 MWt |
| NSSS Thermal Design Flow (per loop) | 80,700 gpm |
| Minimum Measured Flow (per loop) | 87,075 gpm |
| Core Bypass Flow Fraction (non-statistical) (statistical) | 6.5% 5.9% |
| Programmed Full-Power RCS Average Temperature | 572.0°F maximum 549.0°F minimum |
| Steam Generator Design | Westinghouse Model 44F |
| Maximum SGTP Level | 10% average/peak |
| DNB Methodology (where applicable) | RTDP |
| Safety Analysis Limit DNBR | 1.48 (typical & thimble cell) |
| Max $F_{\Delta H}$ (non-statistical) (statistical) | 1.70 1.635 |
| Max F_Q | 2.50 |
| Rod Average Thermal Output | 6.644 kW/ft |
| Initial Condition Uncertainties: | |
| Power | ± 2% RTP |
| RCS flow | ± 4% |
| Temperature | ± 3.6°F, + 3.0°F bias |
| Pressure | ± 25 psi, +3/-12 psi bias |
| Steam generator water level | ± 10% NRS |
| Pressurizer water level | ± 6% span |

| Table 6.3-4 | |
|--|-------------------|
| Sequence of Events-Uncontrolled Rod Withdrawal from Subcritical Event | |
| Event | Time (sec) |
| Start of Accident | 0.0 |
| Power Range High Neutron Flux Low Setpoint Reached | 9.8 |
| Peak Nuclear Power Occurs | 9.9 |
| Rods Begin to Fall into Core | 10.3 |
| Peak Heat Flux Occurs | 11.8 |
| Minimum DNBR Occurs | 11.8 |
| PCT Occurs | 12.0 |
| Peak Average Fuel Temperature Occurs | 12.3 |
| Peak Fuel Centerline Temperature Occurs | 13.2 |

**Table 6.3-5
Sequence of Events-Uncontrolled RCCA Bank Withdrawal
at Power Analysis**

| Case | Event | Time(s) |
|--|---|---------|
| 100% Power, Minimum Feedback, Rapid RCCA Withdrawal (70 pcm/sec) | Initiation of uncontrolled RCCA withdrawal | 0.0 |
| | Power range high neutron flux (high setpoint reached) | 1.6 |
| | Rods begin to fall | 2.1 |
| | Minimum DNBR occurs | 3.0 |
| 100% Power Minimum Feedback, Slow RCCA Withdrawal (1 pcm/sec) | Initiation of uncontrolled RCCA withdrawal | 0.0 |
| | OTΔT setpoint reached | 100.2 |
| | Rods begin to fall | 102.2 |
| | Minimum DNBR occurs | 103.0 |

**Table 6.3-6
Sequence of Events – Loss-of-Load/Turbine-Trip Event**

| Case | Event | Time (Sec) |
|--------------------|---|------------|
| Peak Pressure Case | Loss of electrical load/turbine trip | 0.0 |
| | High-pressurizer pressure reactor trip setpoint reached | 6.25 |
| | Rods begin to drop | 8.25 |
| | Peak-pressurizer pressure occurs | 8.6 |
| Minimum DNBR Case | Loss of electrical load/turbine trip | 0.0 |
| | OTΔT reactor trip setpoint reached | 9.81 |
| | Rods begin to drop | 11.81 |
| | Minimum DNBR occurs | 13.1 |

Table 6.3-7

Time Sequence of Events for Loss-of-Normal Feedwater Flow

| Event | Time (seconds) |
|---|---------------------------|
| Main Feedwater Flow Stops | 20.0 |
| Low-Low Steam Generator Water Level Reactor Trip Setpoint Reached | 64.0 |
| Rods Begin to Drop | 66.0 |
| Automatic AFW Flow from 1 MDAFWP (total 380 gpm) Initiated | 124.0 |
| Operator Action to Establish AFW Flow to Remaining Steam Generators | 666.0 |
| Peak Water Level in the Pressurizer Occurs | 925.0 |

| Table 6.3-8 | |
|---|---------------------------|
| Time Sequence of Events for Loss-of-Non-Emergency AC Power | |
| Event | Time (seconds) |
| Main Feedwater Flow Stops | 20.0 |
| Low-Low Steam Generator Water Level Reactor Trip Setpoint Reached | 64.0 |
| Rods Begin to Drop | 66.0 |
| RCPs Begin to Coast Down | 68.1 |
| Automatic AFW Flow from 1 MDAFW (total 380 gpm) Initiated | 124.0 |
| Operator Action to Establish AFW Flow to Remaining Steam Generators | 666.0 |
| Peak Water Level in the Pressurizer Occurs | 720.0 |

Table 6.3-9

Feedwater System Malfunction at Power – Sequence of Events

| Event | Time (seconds) |
|--|-----------------------|
| 1 Main Feedwater Control Valve Fails Full Open | 0 |
| High-High Steam Generator Water Level Trip Setpoint is Reached | 93.8 |
| Turbine Trip Occurs due to High-High Steam Generator Level | 95.7 |
| Rod Motion Begins | 97.7 |
| Minimum DNBR Occurs | 98.0 |
| Feedwater Isolation Valves Begin to Close | 103.7 |

Table 6.3-10

Time Sequence of Events for the Rupture of a Main Steamline

| Event | Case with Offsite Power Time (sec) | Case without Offsite Power Time (sec) |
|---|--|---|
| Double-Ended Steamline Rupture in Loop 1 (1.4 ft ²) | 0.0 | 0.0 |
| Loss of Offsite Power (RCPs begin coasting down) | -- | 2.99 |
| High Steamline Flow Signal Generated (2/4 loops) | 9.29 | 9.27 |
| Low-Low T _{avg} Signal Reached in Loop 1 | 16.92 | 17.66 |
| Low-Low T _{avg} Signal Reached in Loop 2 | 19.98 | 21.64 |
| Safety Injection, SLI and FWI Actuation due to Coincidence of Low-low T _{avg} (2/4 loops) / High Steam Flow (2/4 loops) ESF | 19.99 | 21.66 |
| MSIV Closure Loops 1, 2, 3, and 4 | 24.89 ⁽¹⁾ | 26.56 ⁽¹⁾ |
| MFIV Closure Loops 1, 2, 3, and 4 | 32.89 ⁽¹⁾ | 34.56 ⁽¹⁾ |
| Peak Core Heat Flux Occurs | 126.24 | 71.49 |

Note:

1. Plus an additional 0.1 second for valve closure time.

Table 6.3-11

**Sequence of Events – Partial Loss-of-Forced
Reactor-Coolant-Flow Event**

| Case | Event | Time (sec) |
|--|-----------------------|-----------------------|
| Partial Loss of Forced Reactor Coolant Flow (4 loops initially operating, 1 loop coasting down) | Coastdown begins | 0.0 |
| | Low flow reactor trip | 1.6 |
| | Rods begin to drop | 2.6 |
| | Minimum DNBR occurs | 3.4 |

Table 6.3-12

**Sequence of Events – Complete Loss-of-Forced
Reactor-Coolant-Flow Event**

| Case | Event | Time (sec) |
|---|--------------------------------------|-----------------------|
| Complete Loss of Forced Reactor Coolant Flow (frequency decay) | Frequency decay begins | 0.0 |
| | Underfrequency trip setpoint reached | 0.6 |
| | Rods begin to drop | 1.6 |
| | Minimum DNBR occurs | 3.6 |

| Table 6.3-13 | | |
|--|-----------------------|------------------------------------|
| Summary of Results for the Locked Rotor/Shaft Break Transient | | |
| Criteria | Analysis Value | Limit |
| Maximum Clad Temperature at Core Hot Spot, °F | 1810 | 2700 |
| Maximum Zr-H ₂ O Reaction at Core Hot Spot, wt. % | 0.31 | 16 |
| Maximum RCS Pressure, psia | 2533 | Faulted Condition Stress Limits |

| Table 6.3-14 | |
|--|-------------------|
| Sequence of Events – Locked Rotor/Shaft Break Transient | |
| Event | Time (sec) |
| Rotor on 1 Pump Locks/Shaft Breaks | 0.0 |
| Low Flow Reactor Trip Setpoint Reached | 0.102 |
| Rods Begin to Drop | 1.102 |
| Maximum Clad Temperature Occurs | 3.8 |
| Maximum RCS Pressure Occurs | 4.9 |

Table 6.3-15

Results of the RCCA Ejection Accident Analysis

| | Beginning of Cycle HFP | Beginning of Cycle HZP | End of Cycle HFP | End of Cycle HZP |
|---|---------------------------------------|---------------------------------------|-----------------------------|-----------------------------|
| Power Level, % | 102 | 0 | 102 | 0 |
| Ejected Rod Worth, %ΔK | 0.17 | 0.65 | 0.20 | 0.79 |
| Delayed Neutron Fraction, % | 0.50 | 0.50 | 0.40 | 0.42 |
| Feedback Reactivity Weighting | 1.46 | 2.16 | 1.5 | 2.96 |
| Trip reactivity, %ΔK | 4.0 | 2.0 | 4.0 | 2.0 |
| F _q before Rod Ejection | 2.5 | -- | 2.5 | -- |
| Ejected rod F _q | 6.8 | 12.0 | 7.1 | 20.0 |
| Number of Operational Pumps | 4 | 2 | 4 | 2 |
| Max Fuel Pellet Average Temperature, °F | 3973 | 2472 | 3861 | 3653 |
| Max Fuel Centerline Temperature, °F | 4959 | 2812 | 4862 | 4018 |
| Max Clad Average Temperature, °F | 2196 | 1881 | 2130 | 2899 |
| Max Fuel Stored Energy, Btu/lb | 311.2 | 178.0 | 300.8 | 281.6 |
| Fuel Melt at the Hot Spot, % | 3.58 | 0 | 3.61 | 0 |

Table 6.3-16**Sequence of Events – RCCA Ejection Accident**

| Case | Event | Time (sec) |
|-----------------|--|-------------------|
| BOL, Full Power | Initiation of rod ejection | 0.0 |
| | Power range high neutron flux setpoint reached | 0.06 |
| | Peak nuclear power occurs | 0.13 |
| | Rods begin to fall | 0.56 |
| | Peak fuel average temperature occurs | 2.17 |
| | PCT occurs | 2.29 |
| | Peak heat flux occurs | 2.31 |
| EOL, Full Power | Initiation of rod ejection | 0.0 |
| | Power range high neutron flux setpoint reached | 0.04 |
| | Peak nuclear power occurs | 0.13 |
| | Rods begin to fall | 0.54 |
| | Peak fuel average temperature occurs | 2.24 |
| | PCT occurs | 2.35 |
| | Peak heat flux occurs | 2.37 |

Table 6.3-17**Sequence of Events – RCCA Ejection Accident**

| Case | Event | Time (sec) |
|-----------------|--|-------------------|
| BOL, Zero Power | Initiation of rod ejection | 0.0 |
| | Power range high neutron flux setpoint reached | 0.34 |
| | Peak nuclear power occurs | 0.40 |
| | Rods begin to fall | 0.84 |
| | PCT occurs | 2.36 |
| | Peak heat flux occurs | 2.37 |
| | Peak fuel average temperature occurs | 2.45 |
| EOL, Zero Power | Initiation of rod ejection | 0.0 |
| | Power range high neutron flux setpoint reached | 0.18 |
| | Peak nuclear power occurs | 0.22 |
| | Rods begin to fall | 0.68 |
| | PCT occurs | 1.65 |
| | Peak heat flux occurs | 1.65 |
| | Peak fuel average temperature occurs | 1.85 |

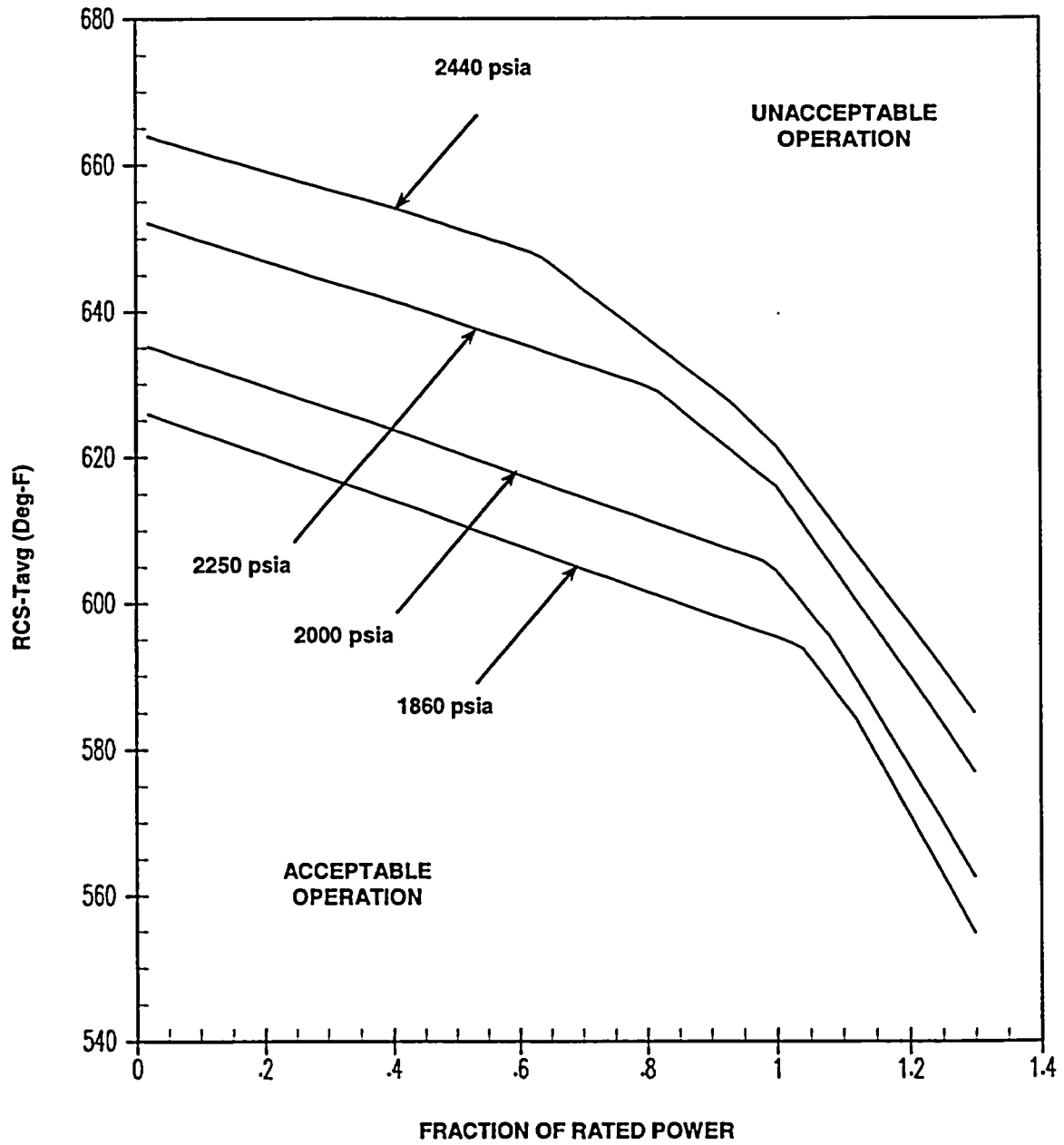


Figure 6.3-1
Reactor Core Safety Limit – Four Loops in Operation

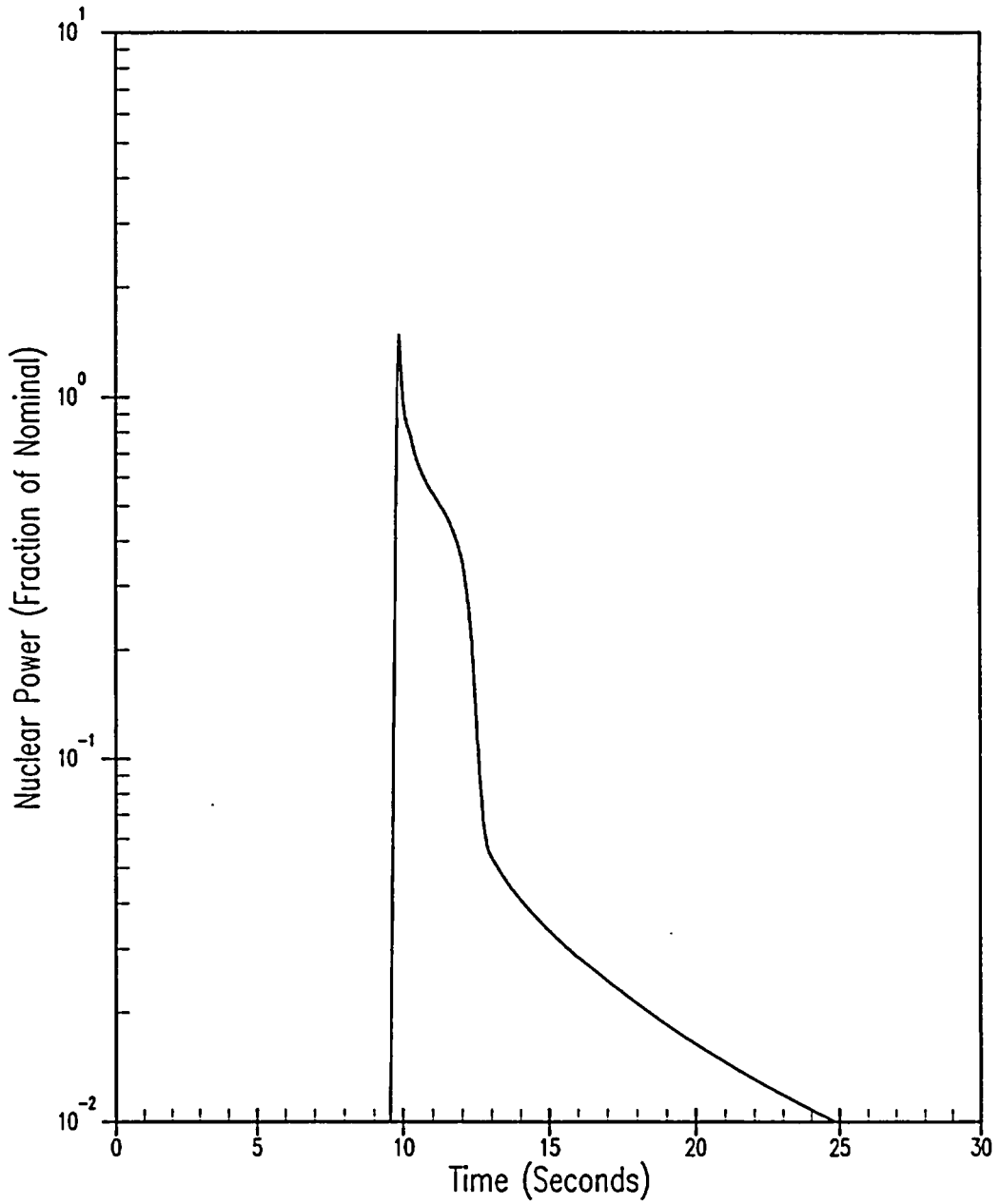


Figure 6.3-2
Neutron Flux Transient for Uncontrolled Rod Withdrawal
from a Subcritical Condition

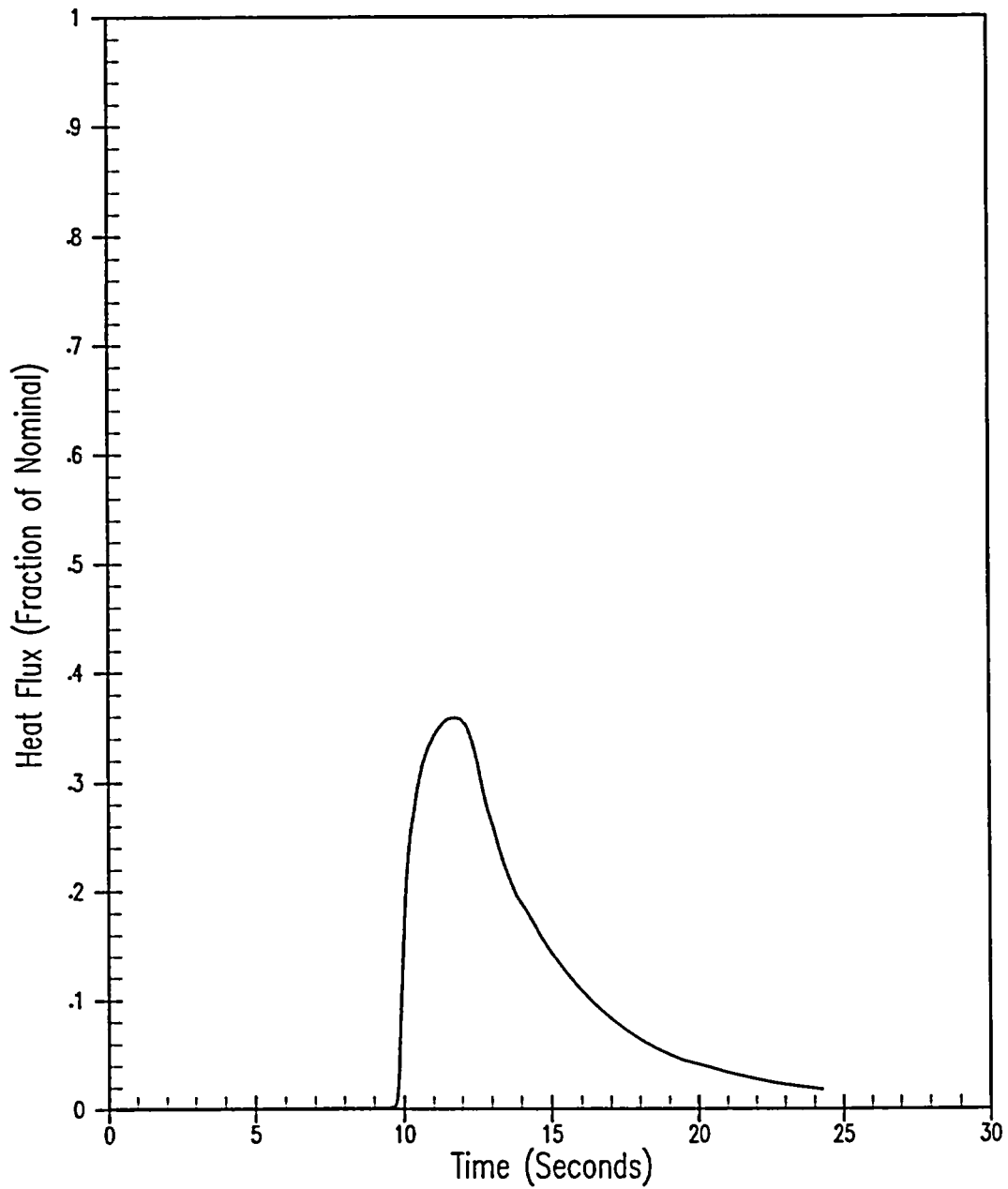


Figure 6.3-3
Thermal Flux Transient for Uncontrolled Rod Withdrawal
from a Subcritical Condition

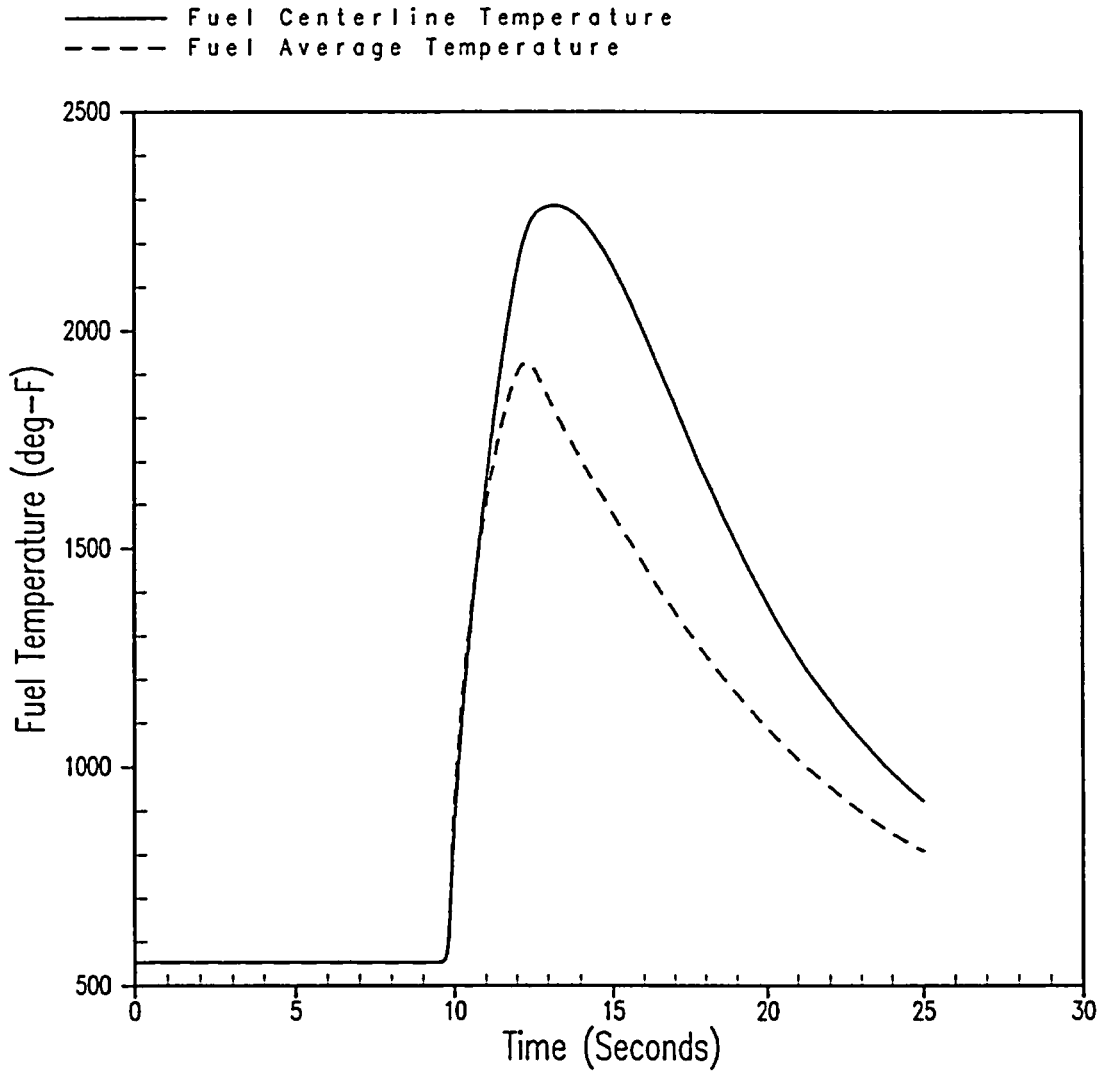


Figure 6.3-4
Fuel Temperature Transient for Uncontrolled Rod Withdrawal
from a Subcritical Condition (Temperature vs. Time)

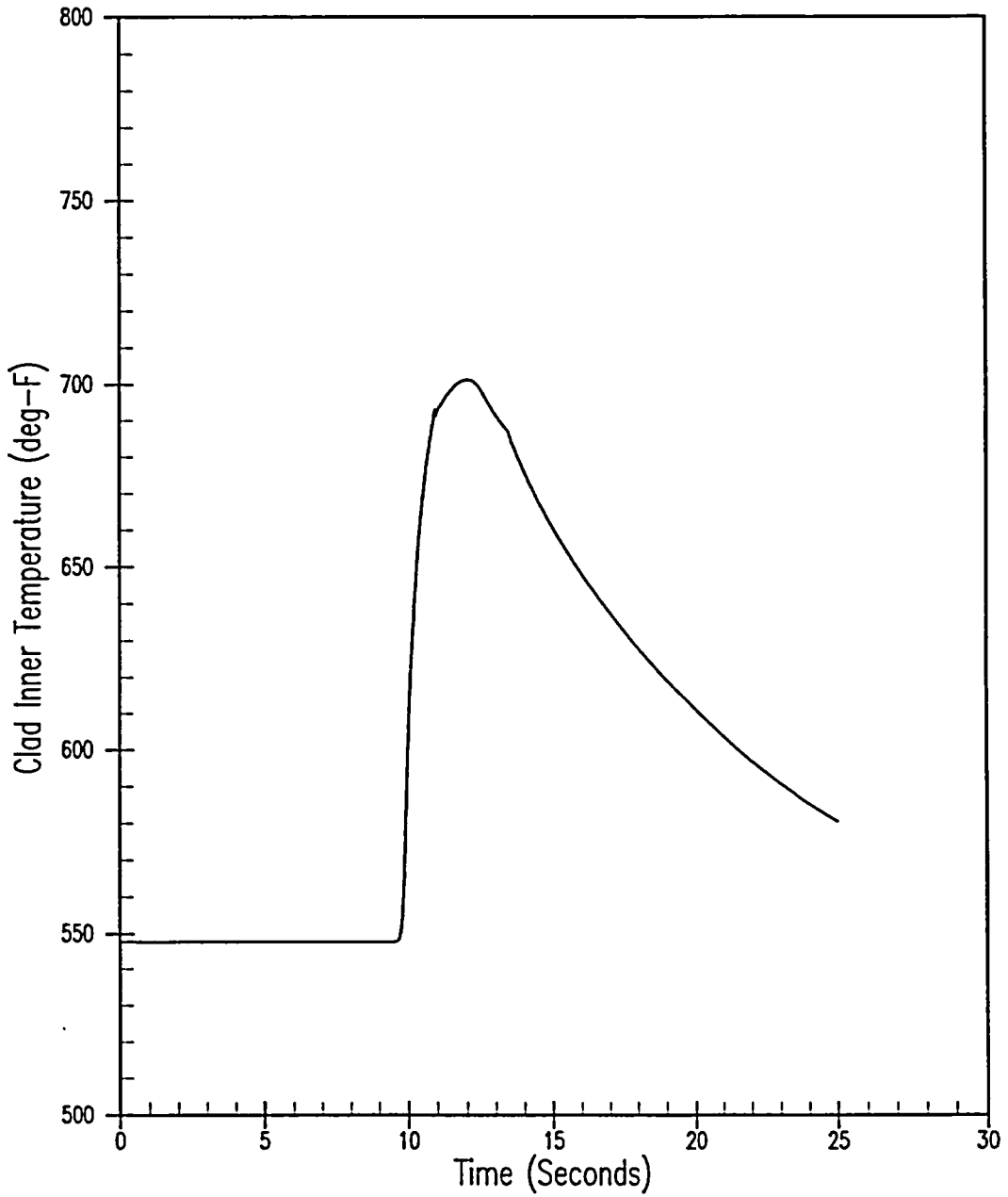


Figure 6.3-5
Clad Inner Temperature Transient for Uncontrolled Rod Withdrawal
from a Subcritical Condition

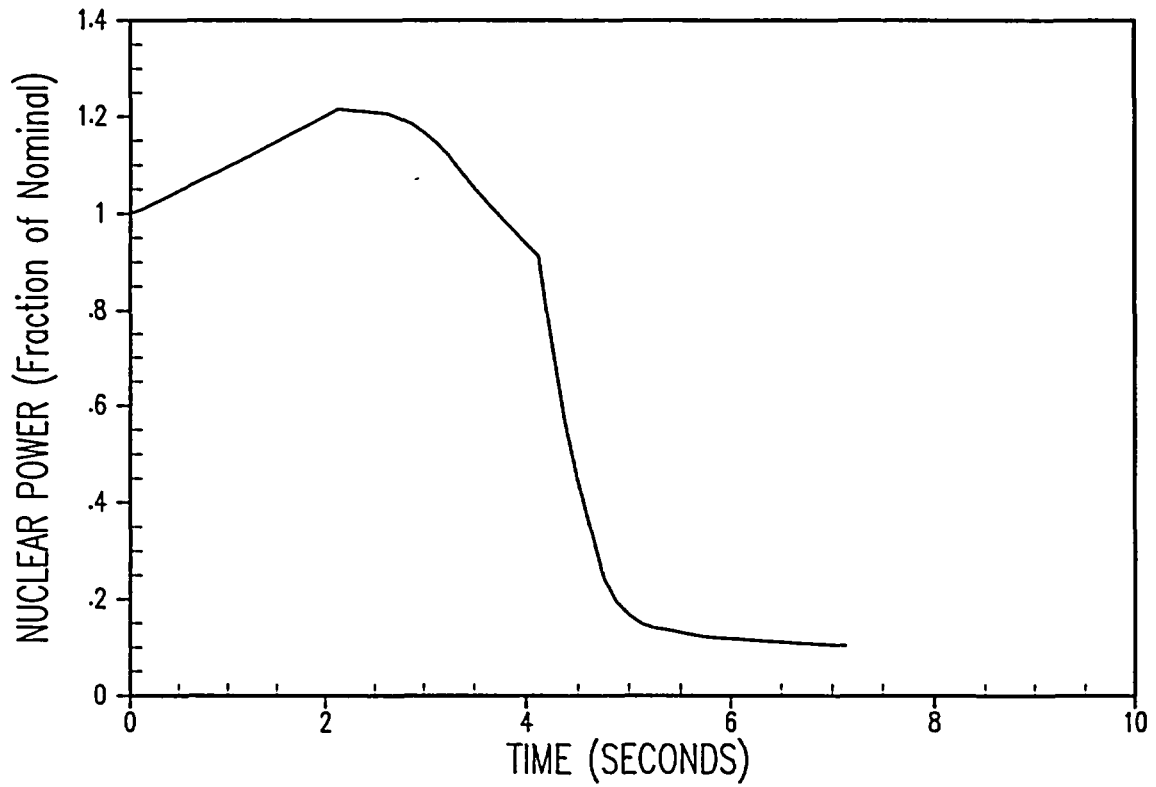


Figure 6.3-6
Uncontrolled RCCA Bank Withdrawal at Power Withdrawal Rate of 70 pcm/second
100% Power - Minimum Reactivity Feedback (Nuclear Power vs. Time)

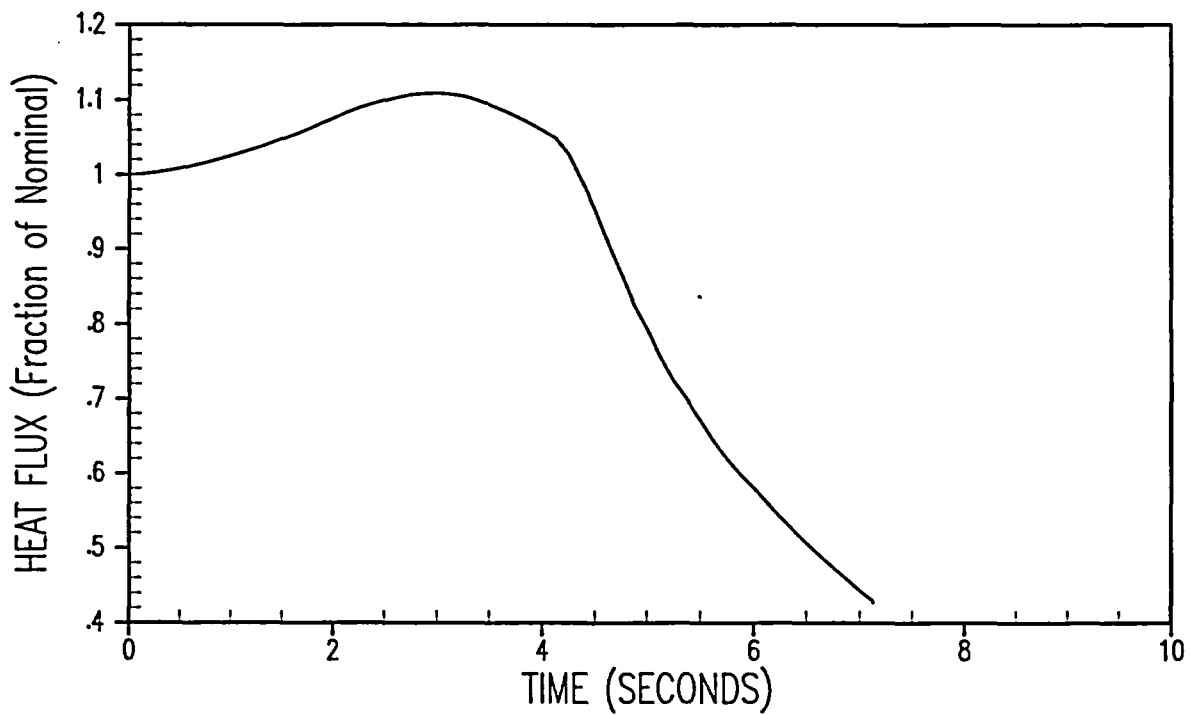


Figure 6.3-7

**Uncontrolled RCCA Bank Withdrawal at Power Withdrawal Rate of 70 pcm/second
100% Power - Minimum Reactivity Feedback (Core Heat Flux vs. Time)**

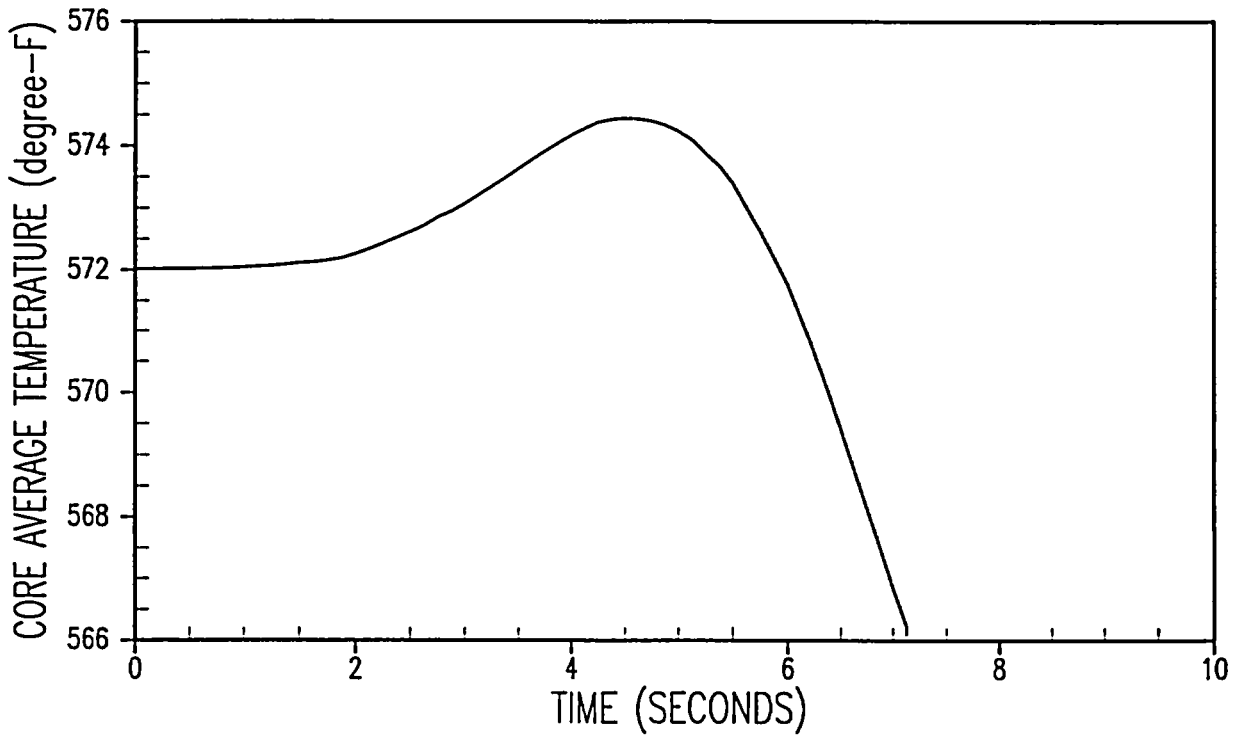


Figure 6.3-8

**Uncontrolled RCCA Bank Withdrawal at Power Withdrawal Rate of 70 pcm/second
100% Power - Minimum Reactivity Feedback (Core Average Temperature vs. Time)**

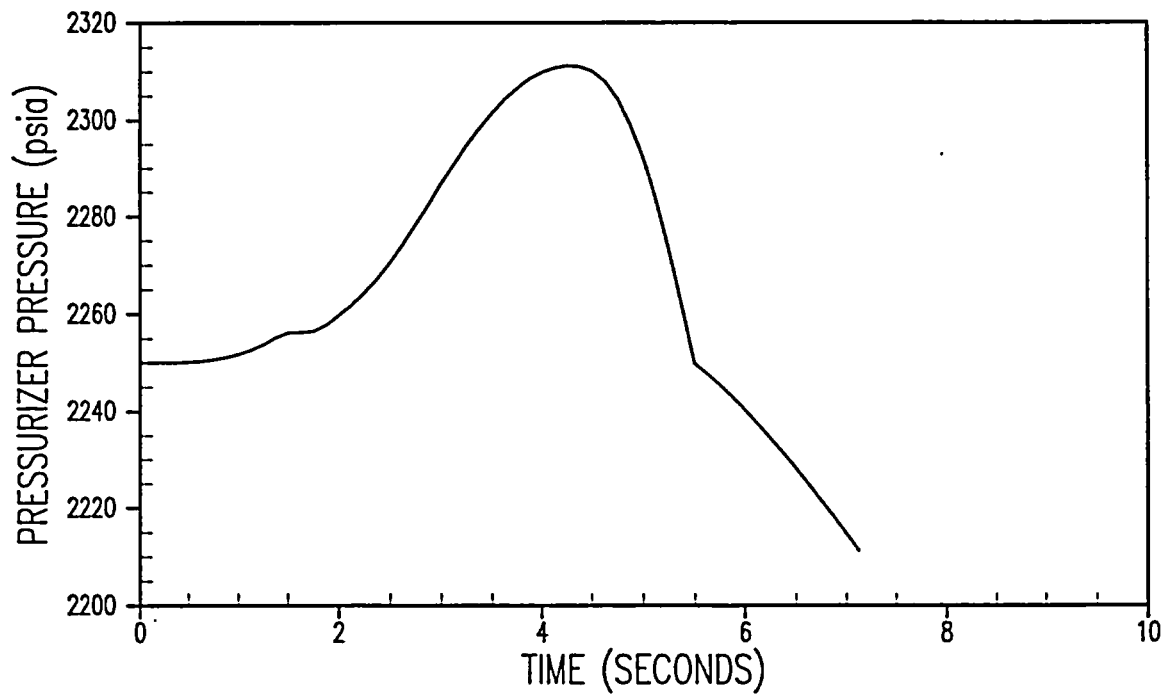


Figure 6.3-9

**Uncontrolled RCCA Bank Withdrawal at Power Withdrawal Rate of 70 pcm/second
100% Power - Minimum Reactivity Feedback (Pressurizer Pressure vs. Time)**

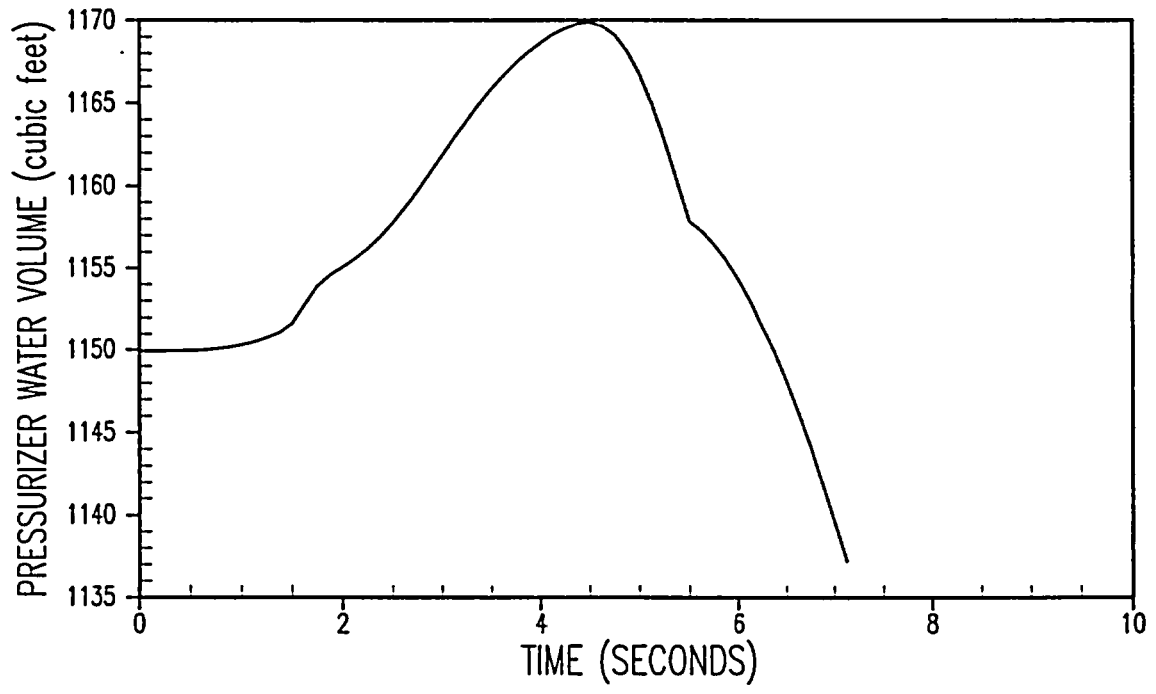


Figure 6.3-10
Uncontrolled RCCA Bank Withdrawal at Power Withdrawal Rate of 70 pcm/second
100% Power - Minimum Reactivity Feedback (Pressurizer Water Volume vs. Time)

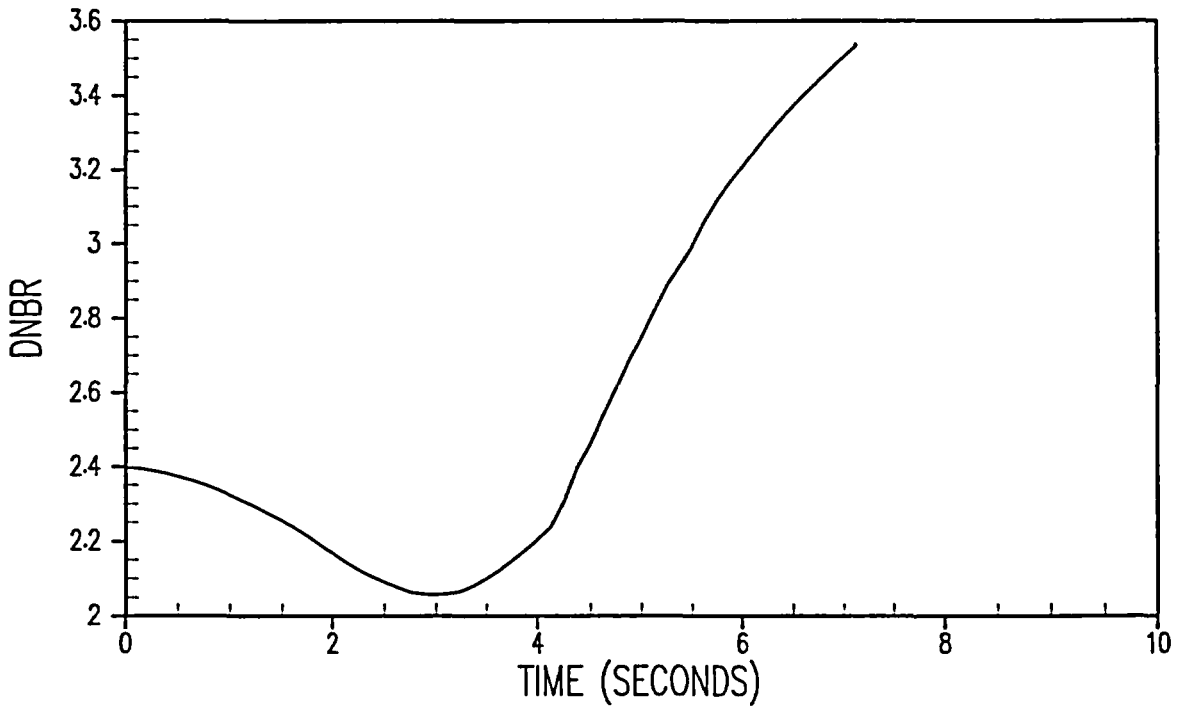


Figure 6.3-11

**Uncontrolled RCCA Bank Withdrawal at Power Withdrawal Rate of 70 pcm/second
100% Power - Minimum Reactivity Feedback (DNBR vs. Time)**

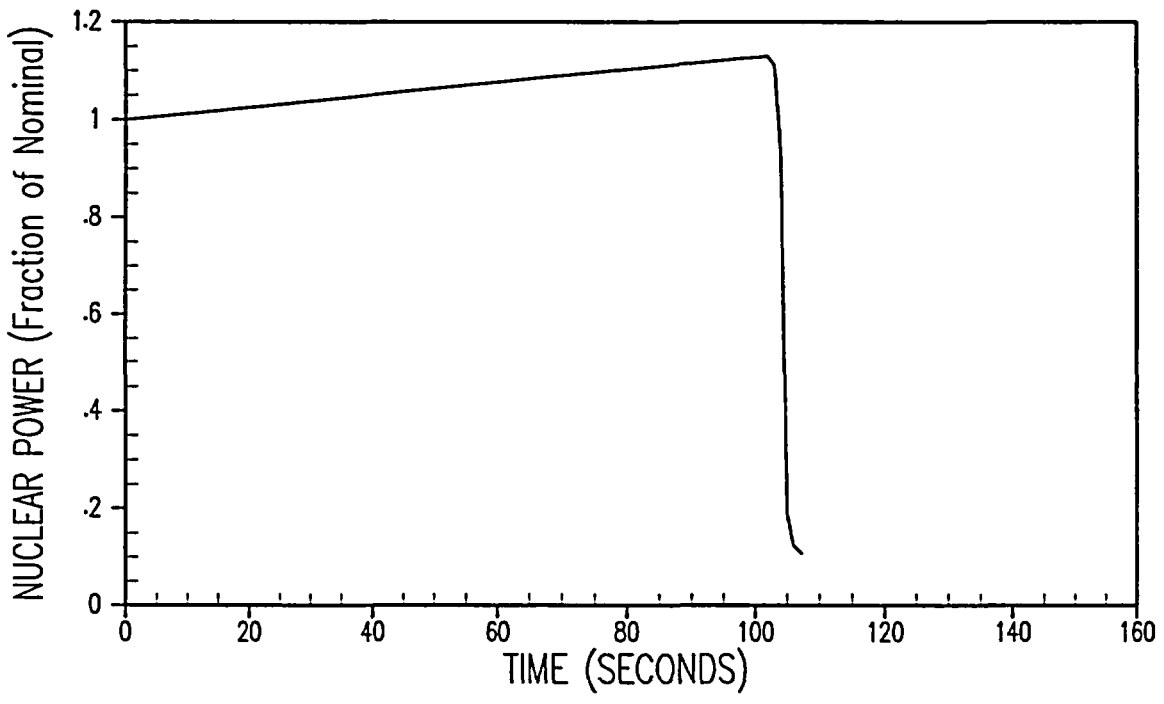


Figure 6.3-12
Uncontrolled RCCA Bank Withdrawal at Power Withdrawal Rate of 1 pcm/second
100% Power - Minimum Reactivity Feedback (Nuclear Power vs. Time)

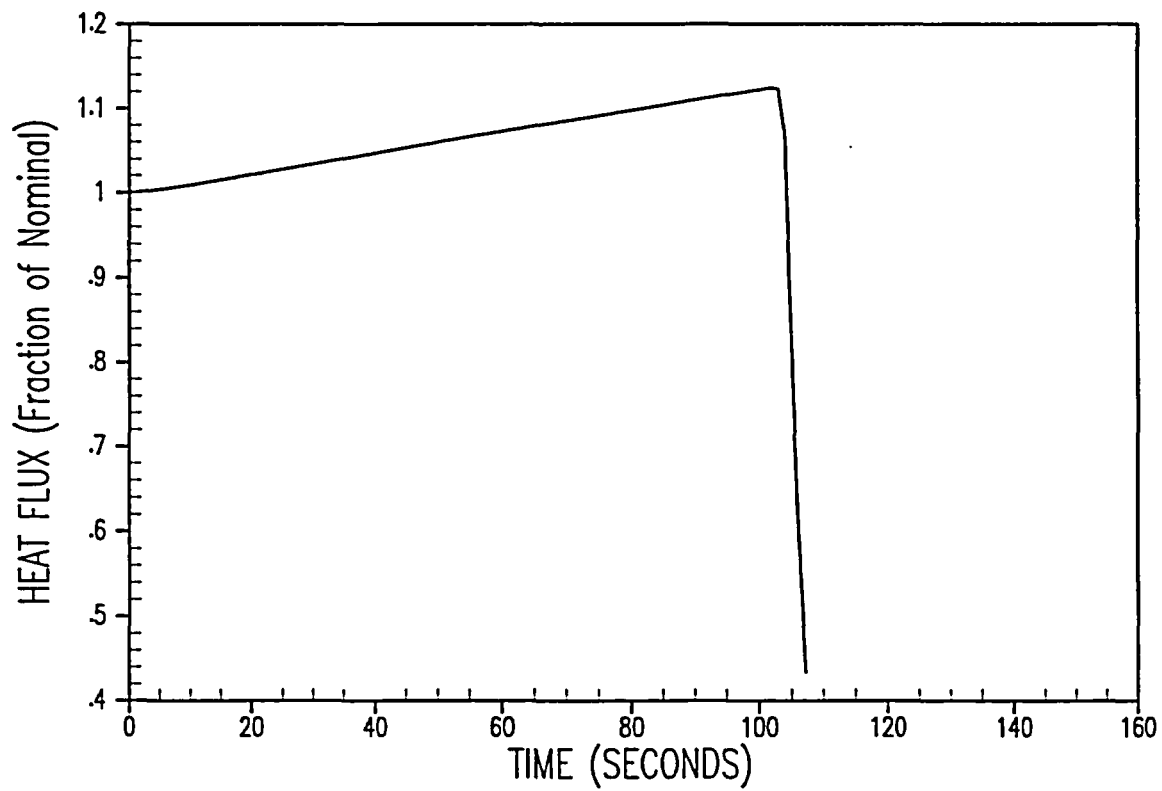


Figure 6.3-13

**Uncontrolled RCCA Bank Withdrawal at Power Withdrawal Rate of 1 pcm/second
100% Power - Minimum Reactivity Feedback (Core Heat Flux vs. Time)**

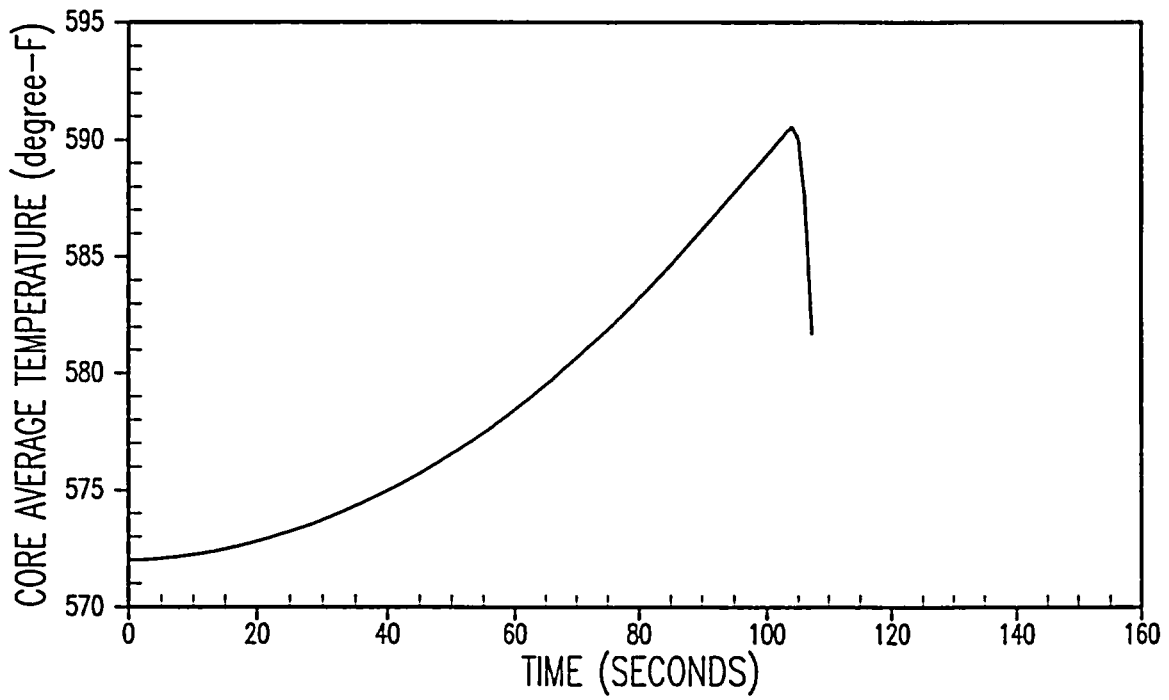


Figure 6.3-14
Uncontrolled RCCA Bank Withdrawal at Power Withdrawal Rate of 1 pcm/second
100% Power - Minimum Reactivity Feedback (Core Average Temperature vs. Time)

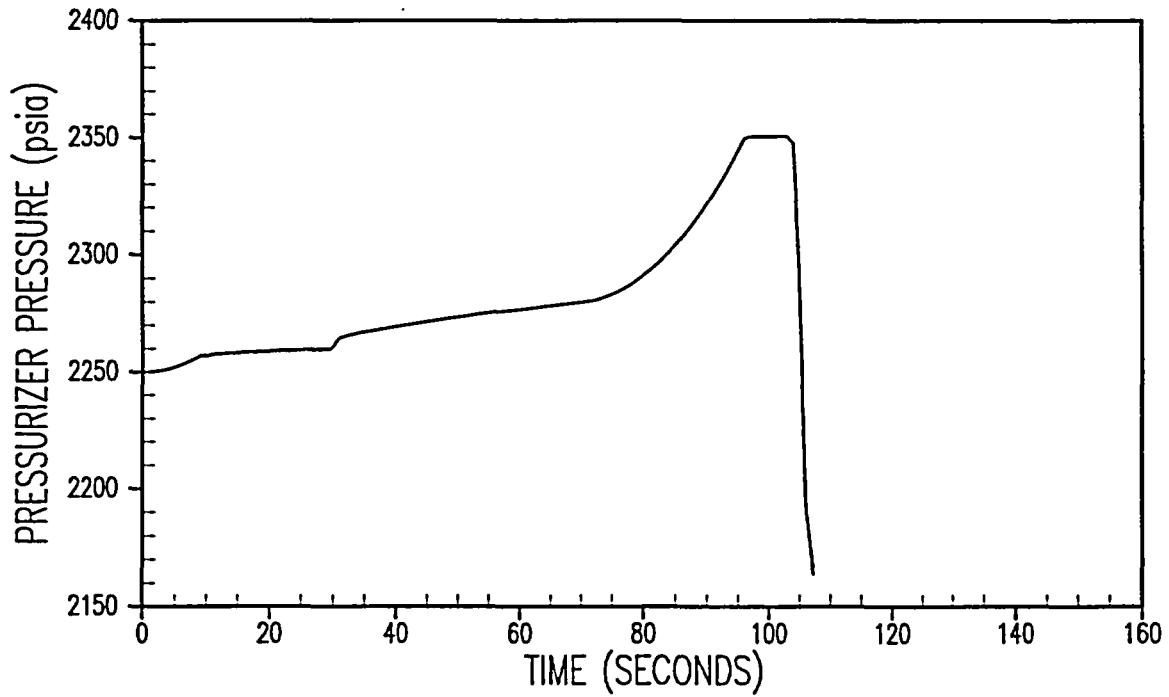


Figure 6.3.3-15

**Uncontrolled RCCA Bank Withdrawal at Power Withdrawal Rate of 1 pcm/second
100% Power - Minimum Reactivity Feedback (Pressurizer Pressure vs. Time)**

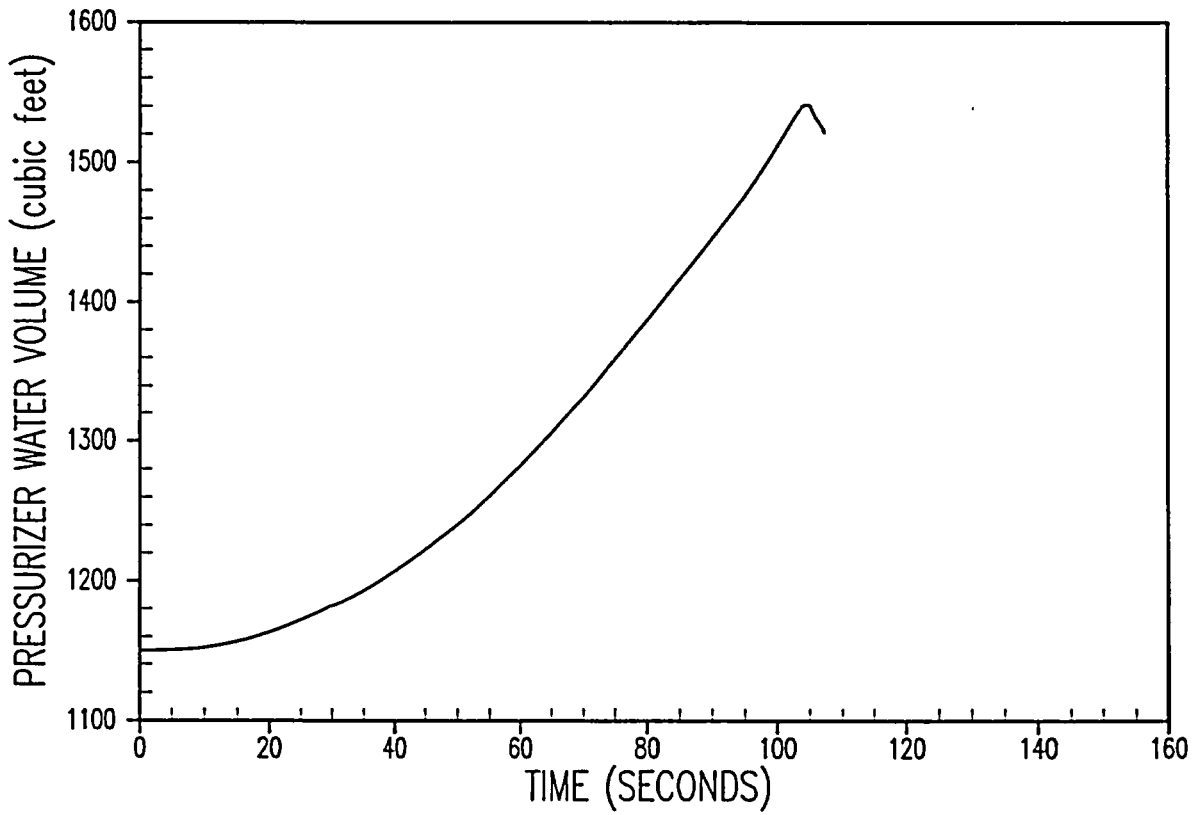


Figure 6.3-16

**Uncontrolled RCCA Bank Withdrawal at Power Withdrawal Rate of 1 pcm/second
100% Power - Minimum Reactivity Feedback (Pressurizer Water Volume vs. Time)**

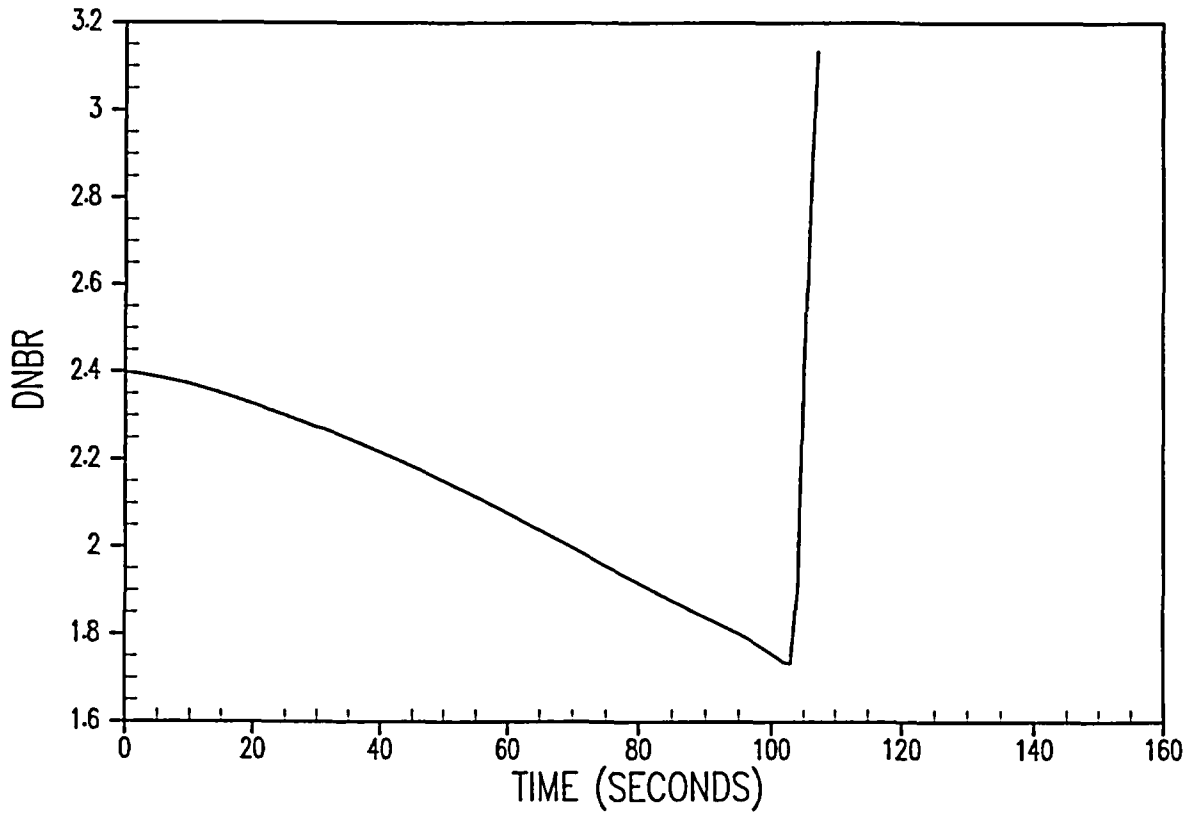


Figure 6.3-17
Uncontrolled RCCA Bank Withdrawal at Power Withdrawal Rate of 1 pcm/second
100% Power - Minimum Reactivity Feedback (DNBR vs. Time)

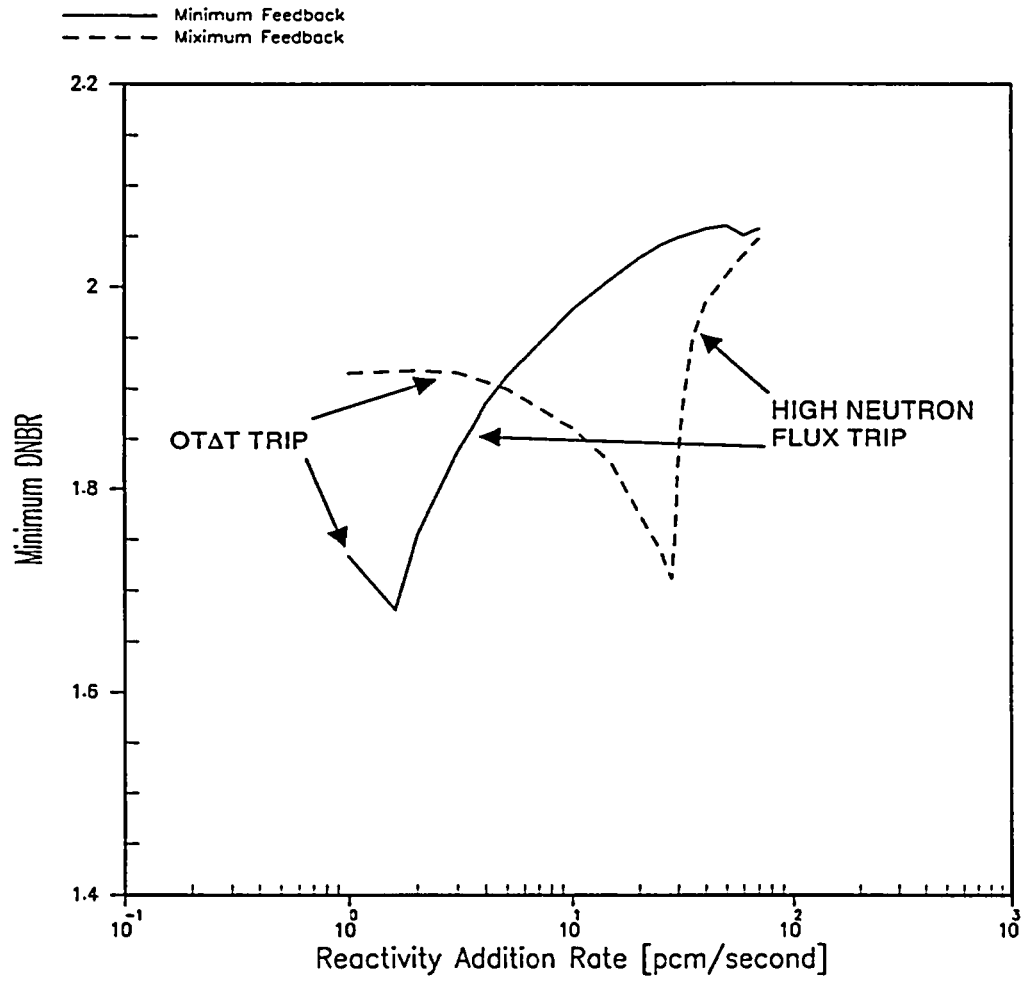


Figure 6.3-18
Uncontrolled RCCA Bank Withdrawal at 100% Power
Minimum DNBR versus Reactivity Insertion Rate

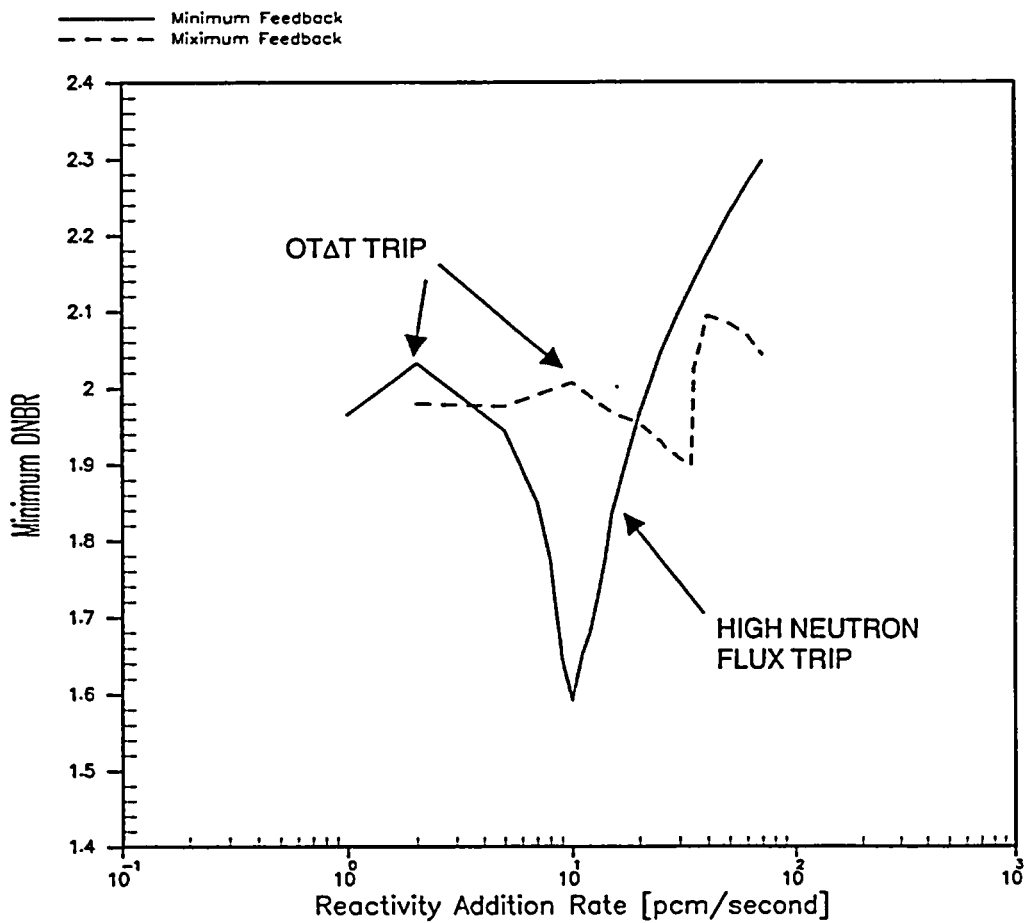


Figure 6.3.3-19
Uncontrolled RCCA Bank Withdrawal at 60% Power
Minimum DNBR vs. Reactivity Insertion Rate

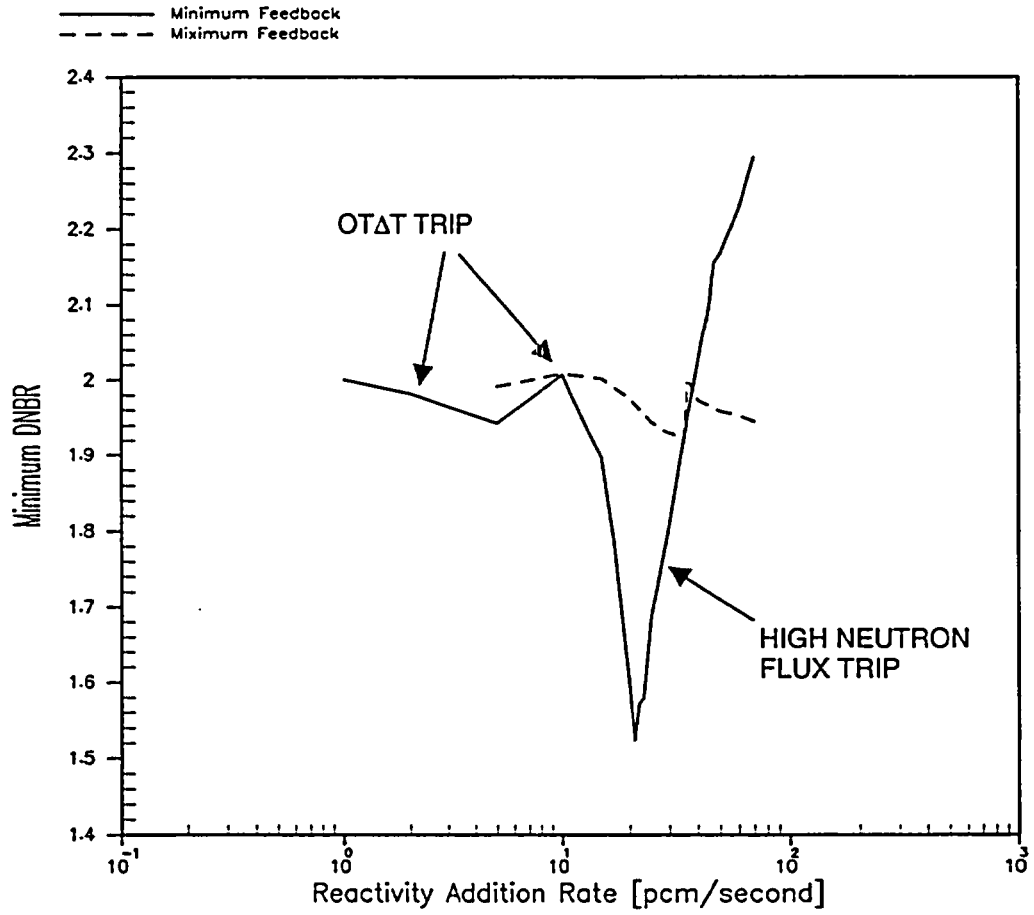


Figure 6.3-20
Uncontrolled RCCA Bank Withdrawal at 10% Power
Minimum DNBR vs. Reactivity Insertion Rate

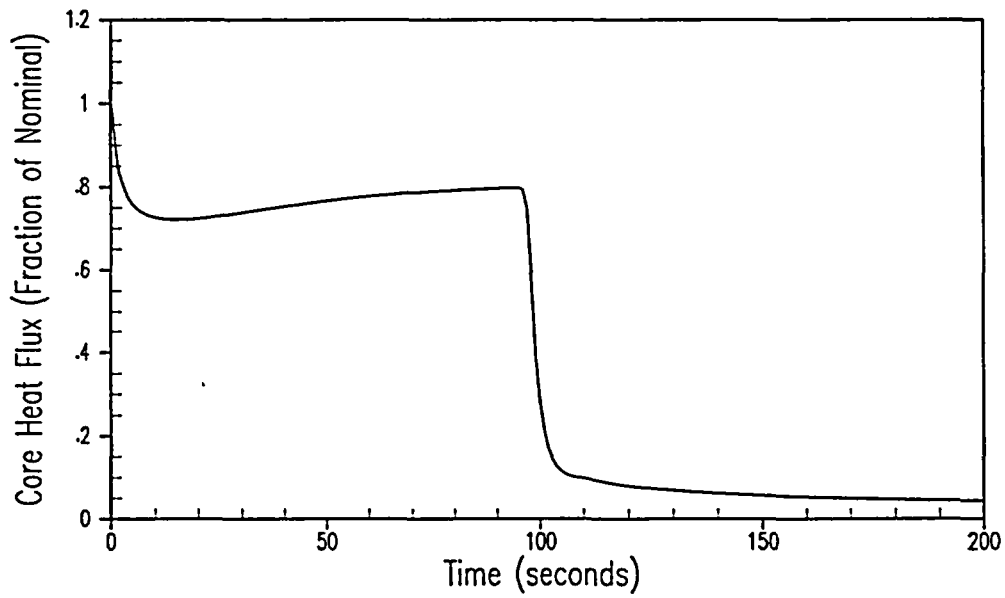
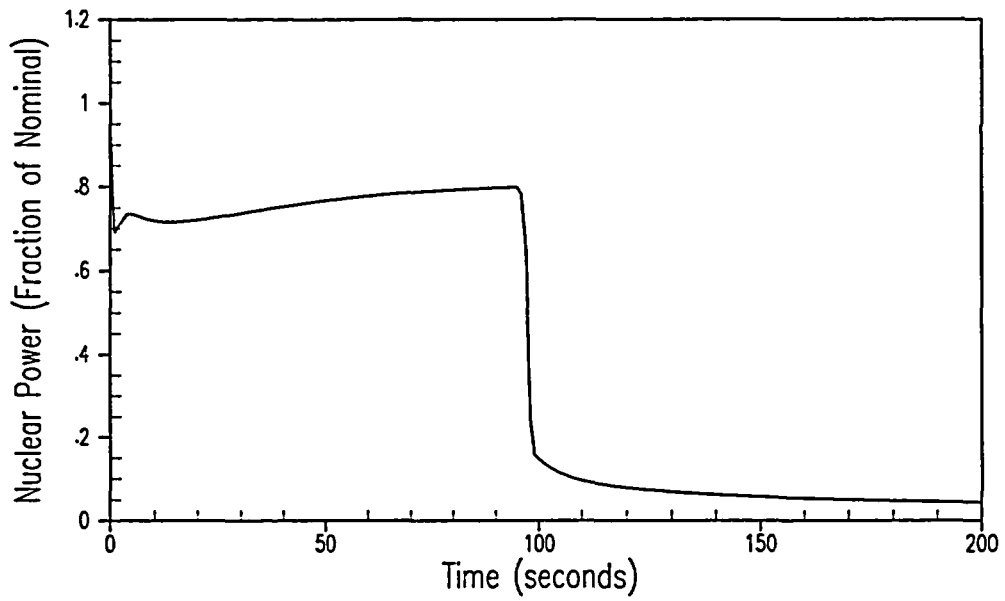


Figure 6.3-21
Dropped Rod Transient with Manual Rod Control, Nuclear Power and Core Heat Flux for
Dropped RCCA Worth of 400 pcm at BOL (small negative MTC)

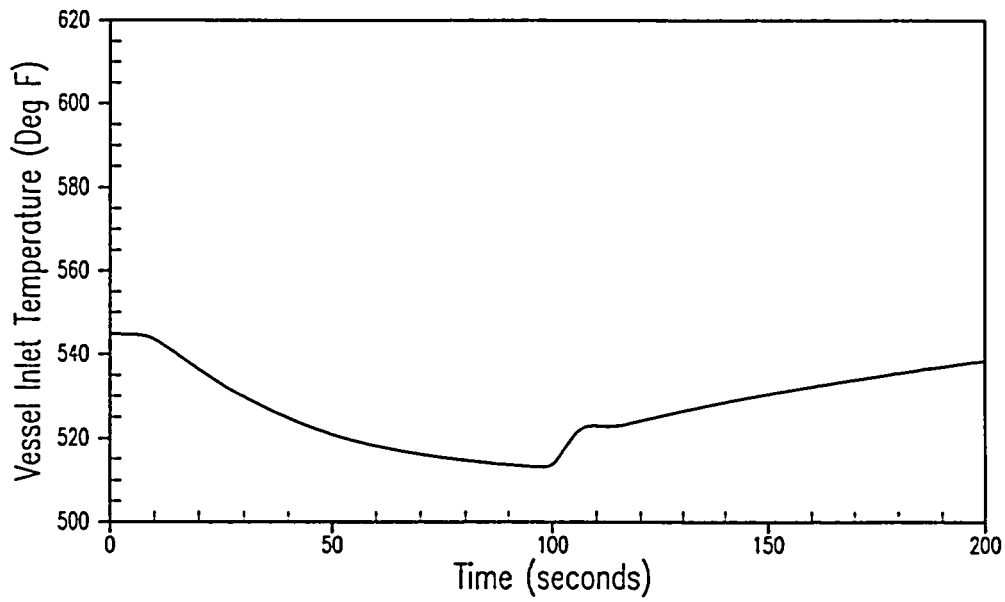
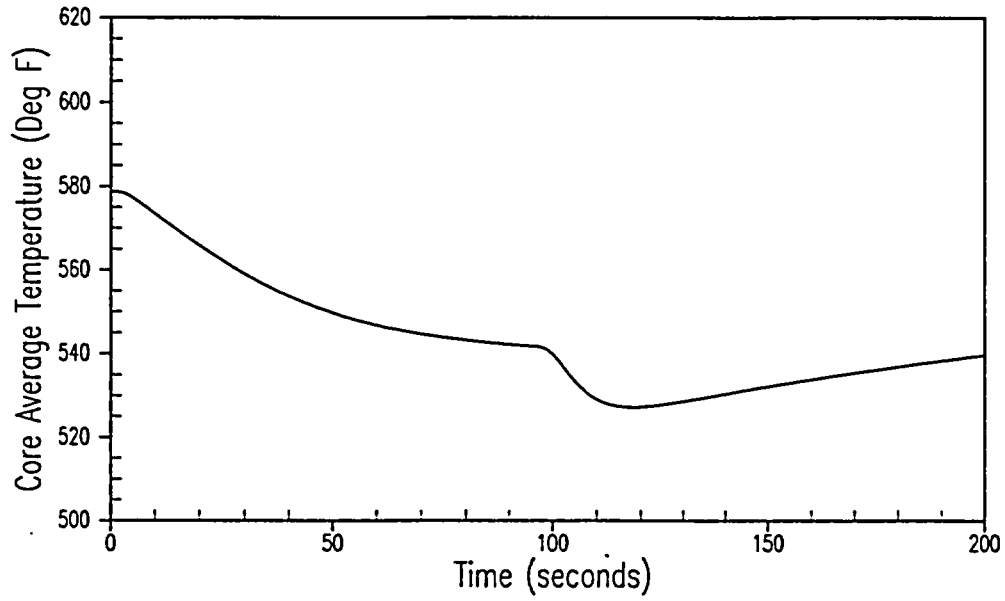


Figure 6.3-22
Dropped Rod Transient with Manual Rod Control, Core Average and Vessel Inlet
Temperature for Dropped RCCA Worth of 400 pcm at BOL (small negative MTC)

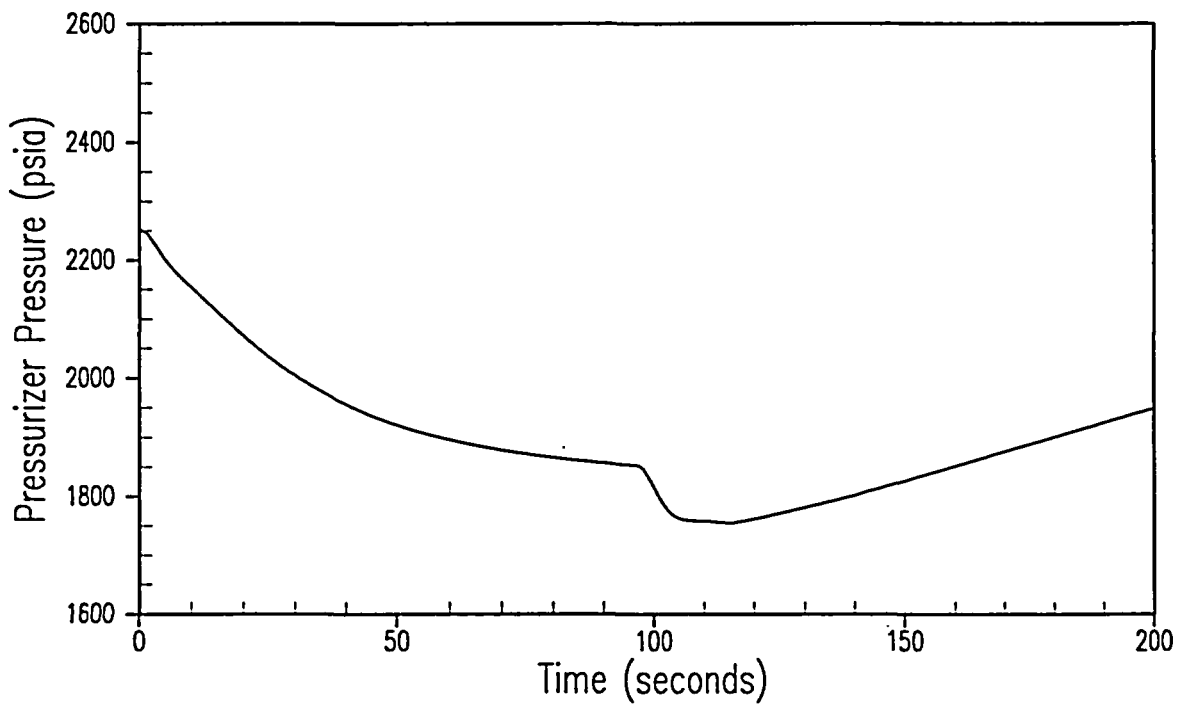


Figure 6.3-23
Dropped Rod Transient with Manual Rod Control, Pressurizer Pressure for Dropped
RCCA Worth of 400 pcm at BOL (small negative MTC)

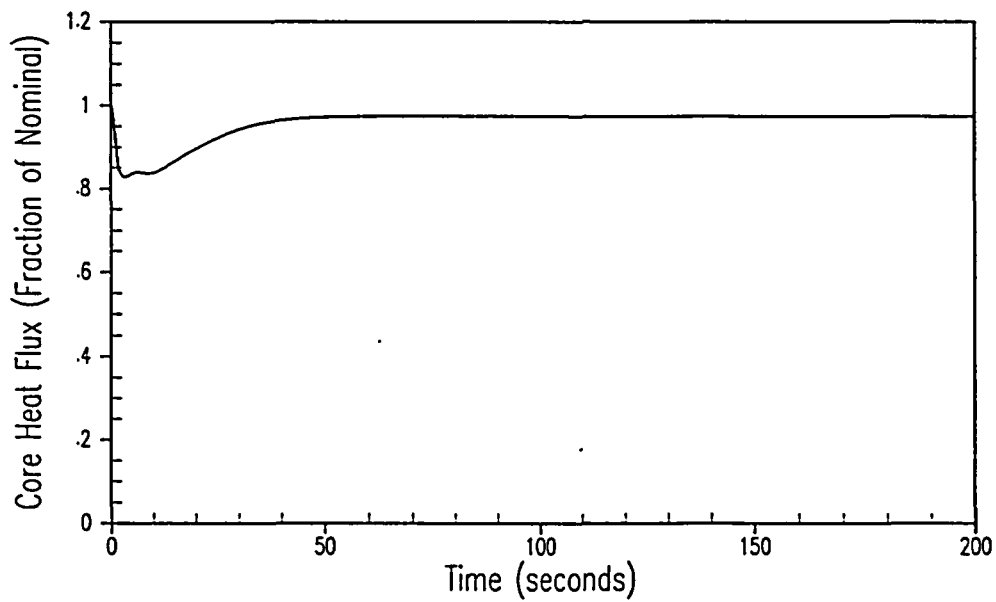
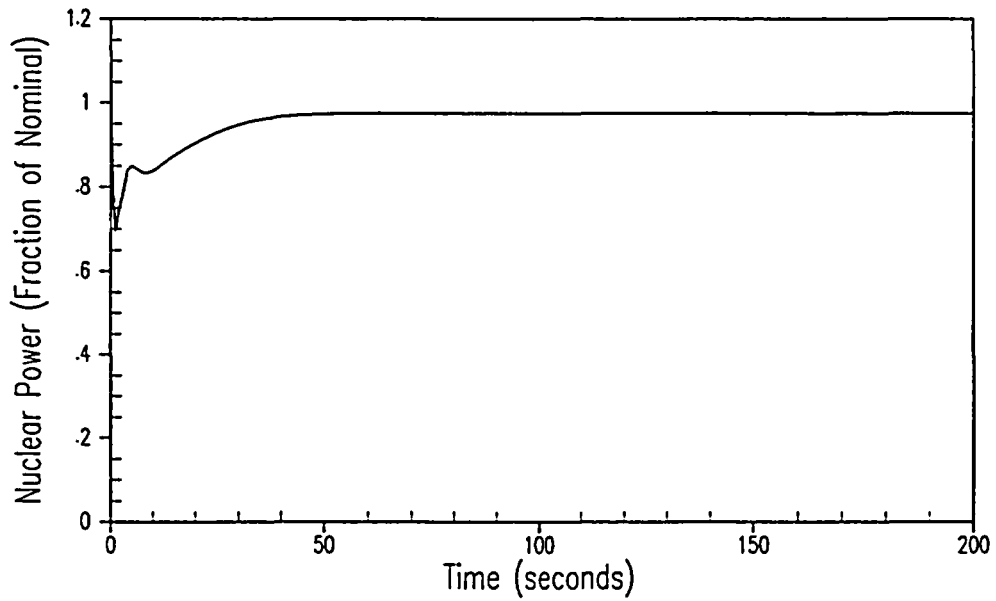


Figure 6.3-24
Dropped Rod Transient with Manual Rod Control, Nuclear Power and Core Heat Flux for
Dropped RCCA Worth of 400 pcm at EOL (large negative MTC)

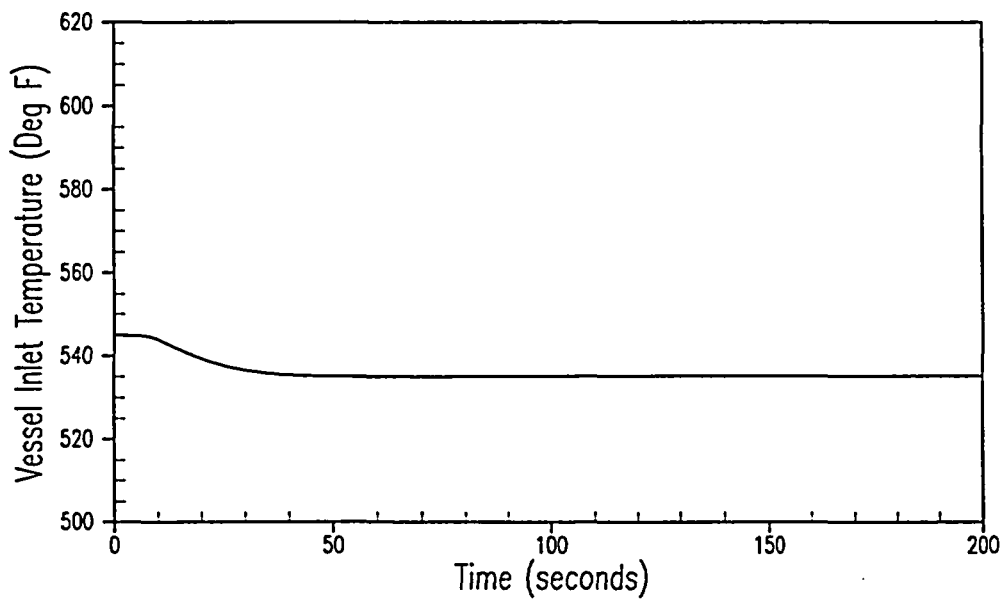
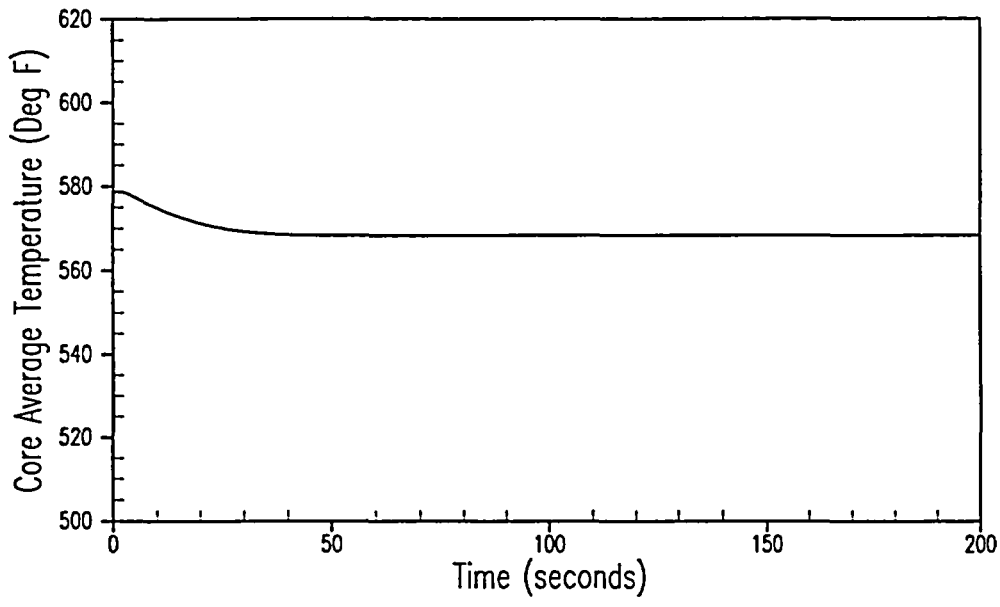


Figure 6.3-25
Dropped Rod Transient with Manual Rod Control, Core Average and Vessel Inlet Temperature for Dropped RCCA Worth of 400 pcm at EOL (large negative MTC)

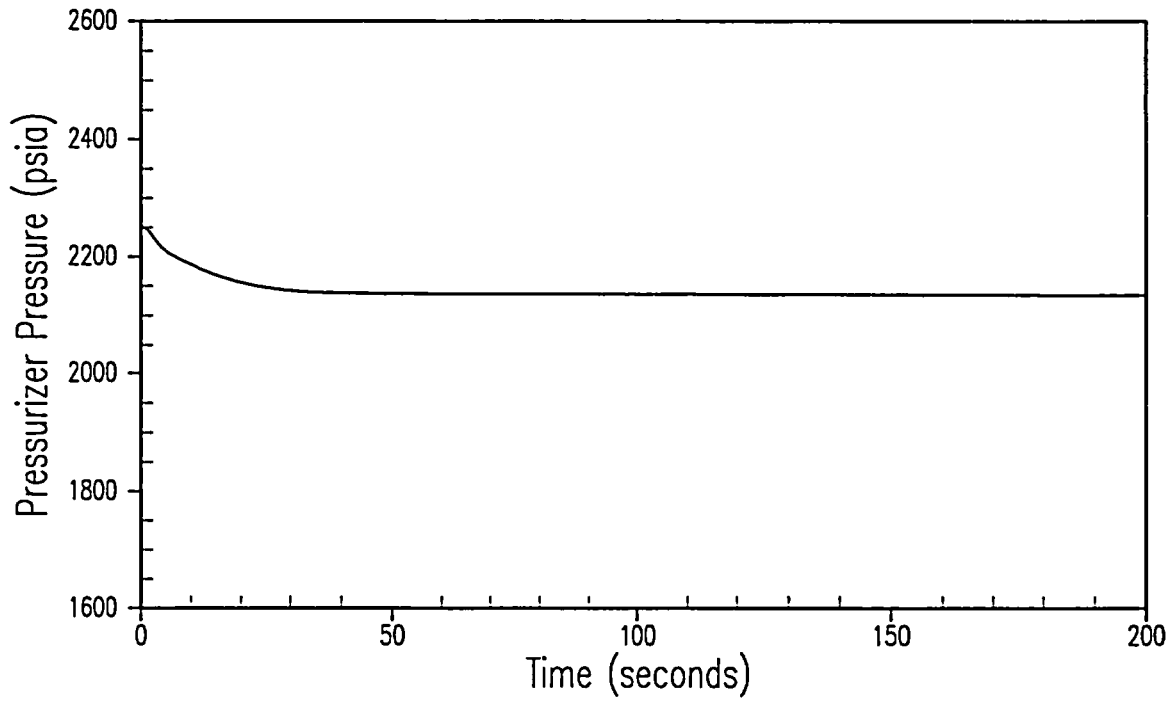


Figure 6.3-26
Dropped Rod Transient with Manual Rod Control, Pressurizer Pressure for Dropped
RCCA Worth of 400 pcm at EOL (large negative MTC)

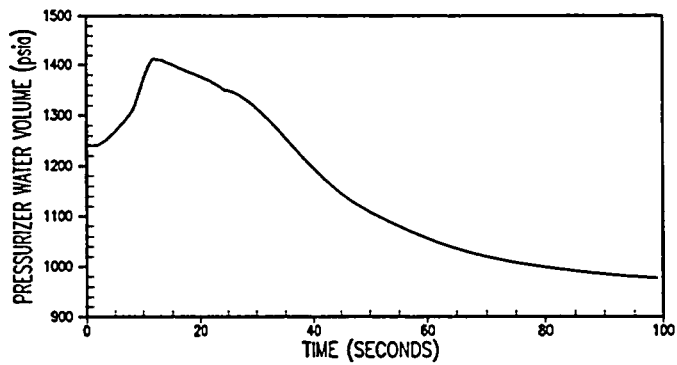
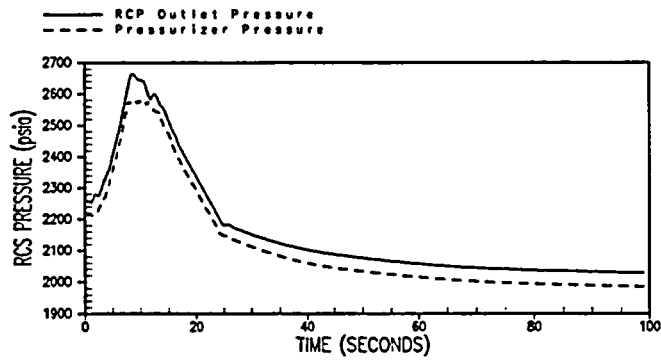
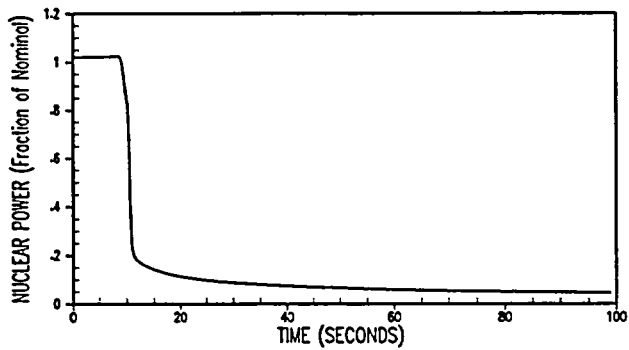


Figure 6.3-27
Loss-of-Load/Turbine Trip, Peak Pressure Case

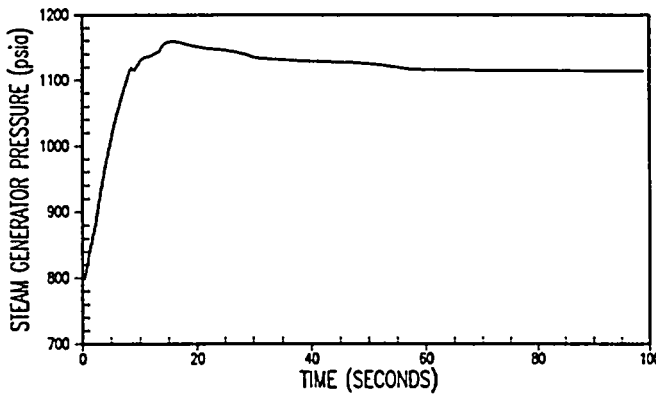
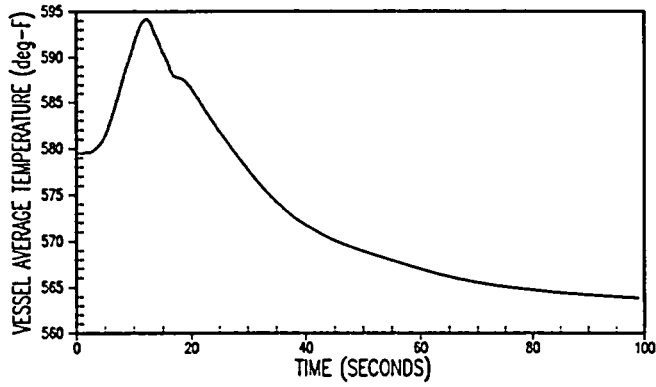
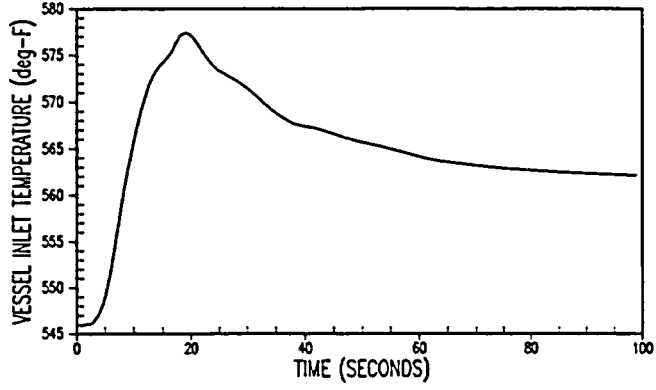


Figure 6.3-28
Loss-of-Load/Turbine Trip, Peak Pressure Case

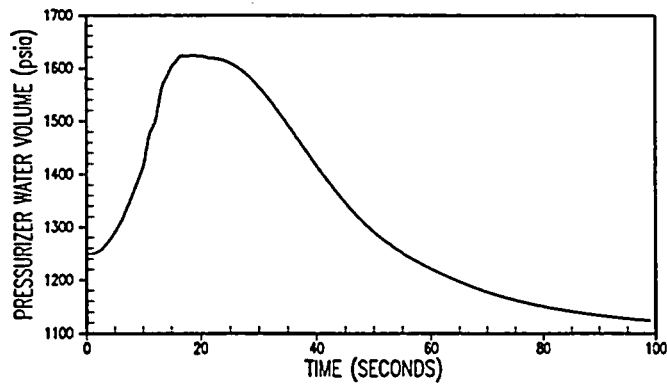
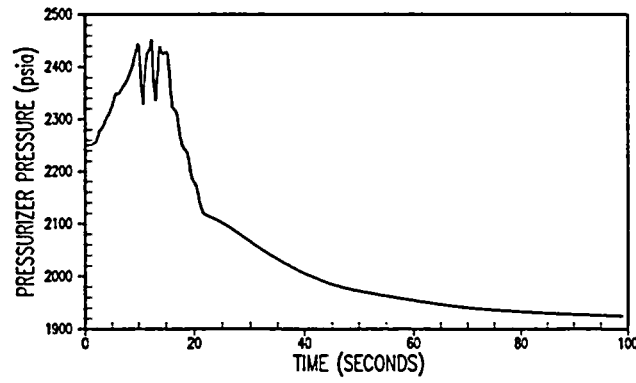
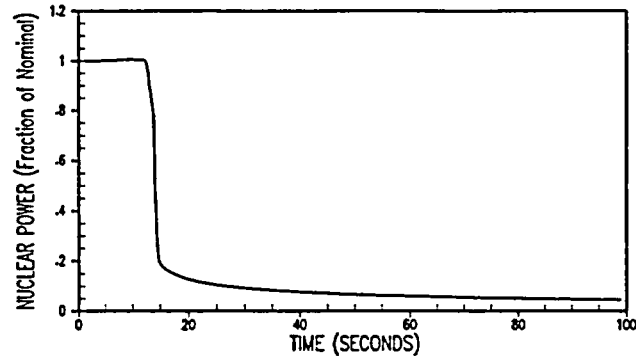


Figure 6.3-29
Loss-of-Load/Turbine Trip, Minimum DNBR Case

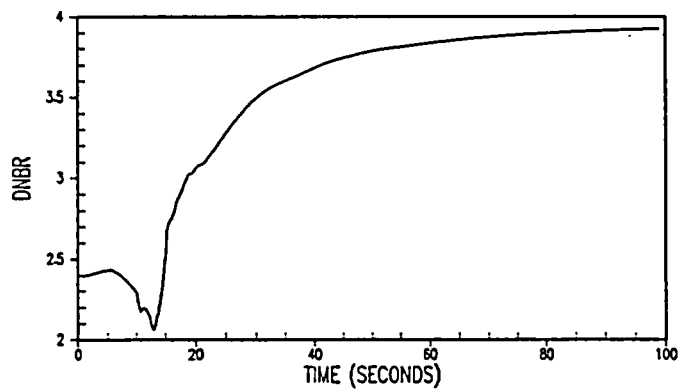
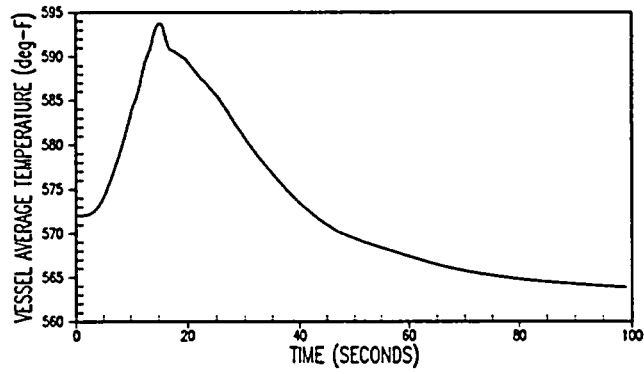
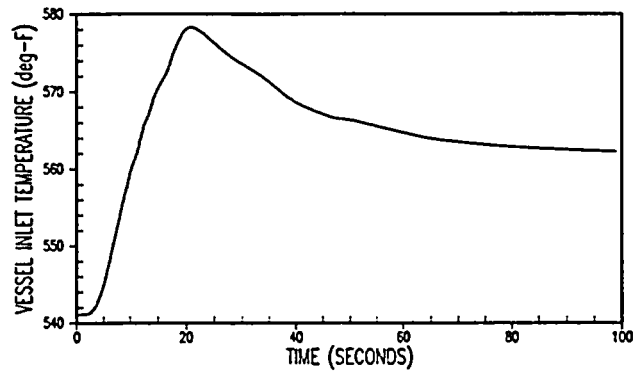


Figure 6.3-30
Loss-of-Load/Turbine Trip, Minimum DNBR Case

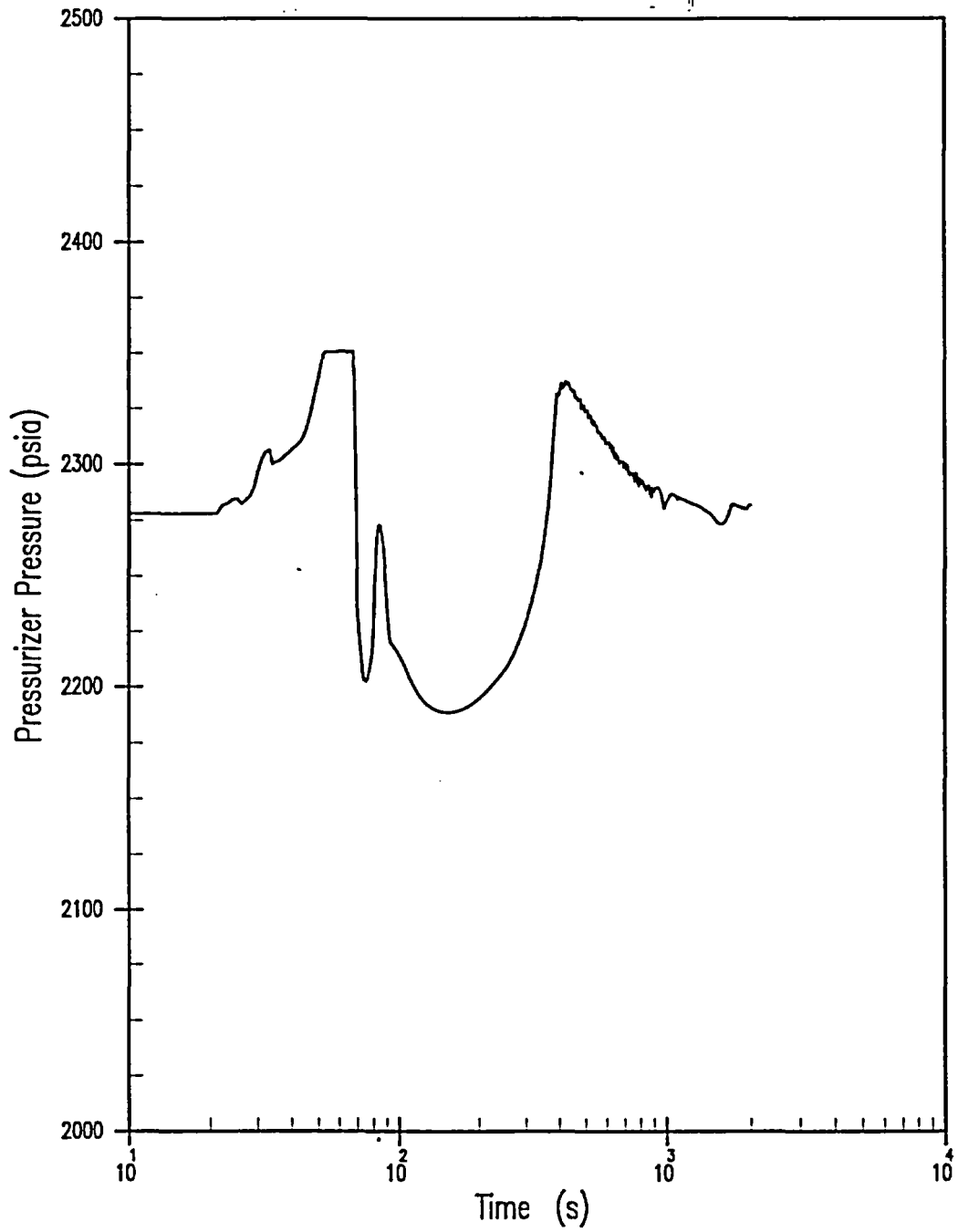


Figure 6.3-31
LONF Pressurizer (Pressure vs. Time)

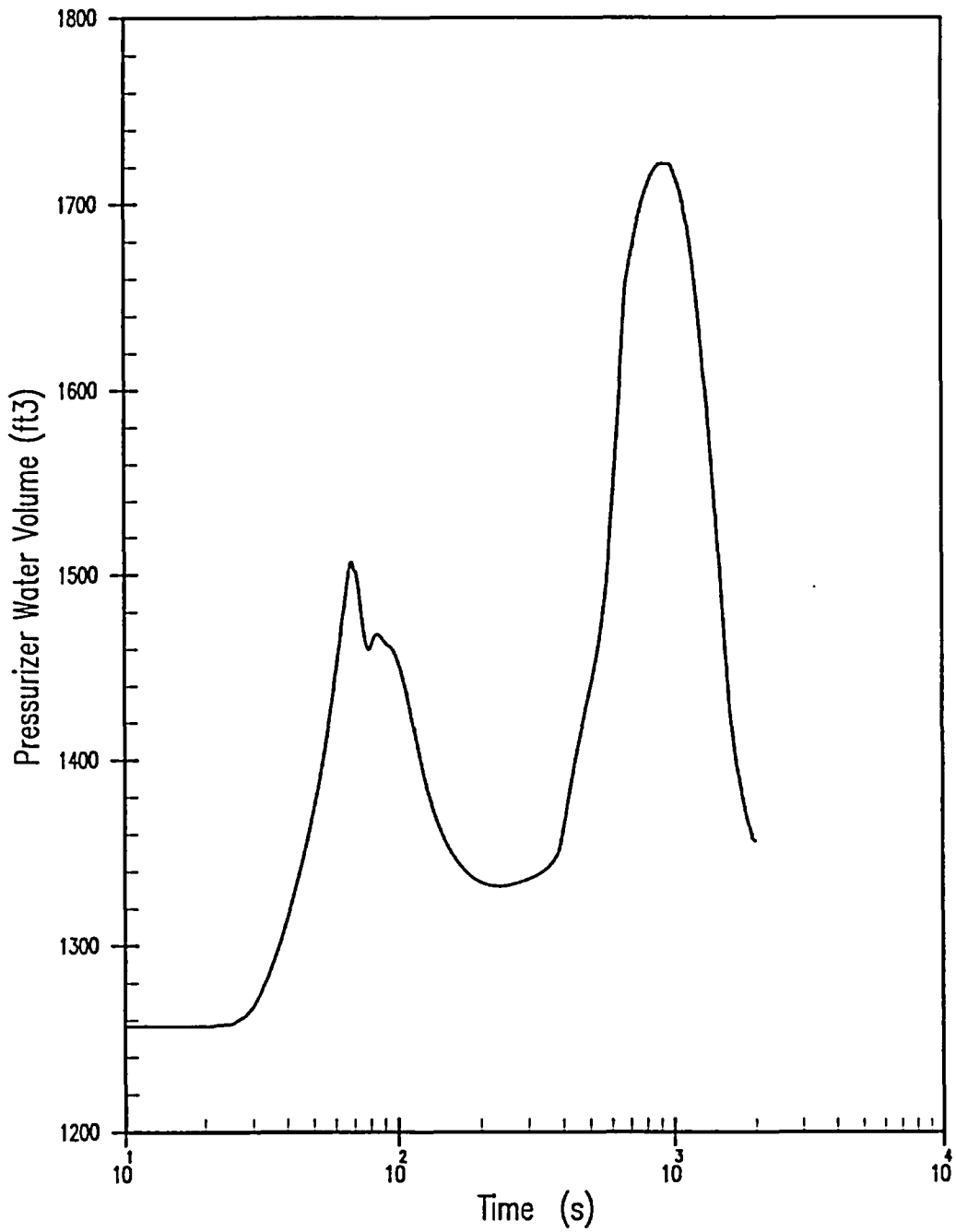


Figure 6.3-32
LONF Pressurizer (Water Volume vs. Time)

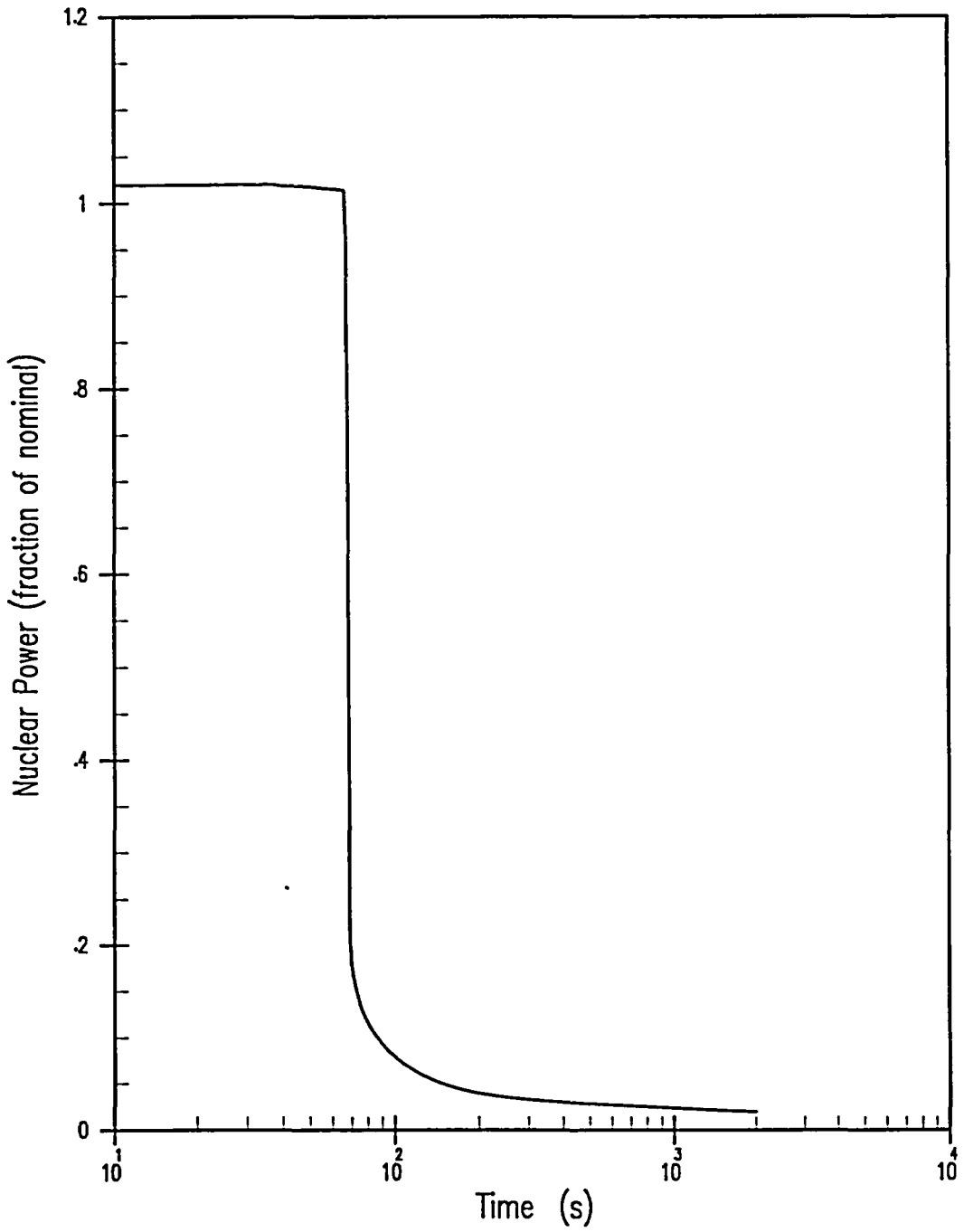


Figure 6.3-33
LONF (Nuclear Power vs. Time)

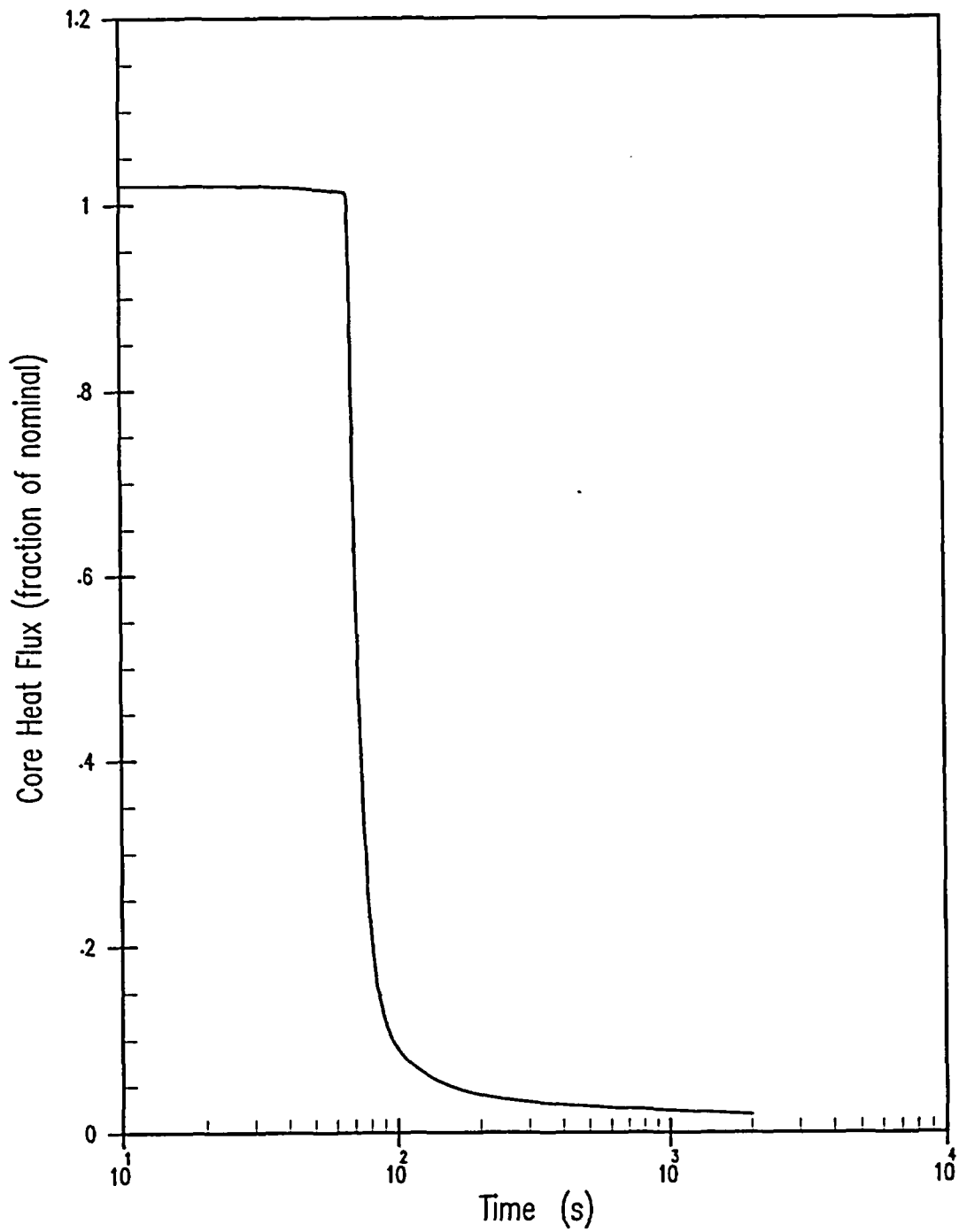


Figure 6.3-34
LONF (Core Heat Flux vs. Time)

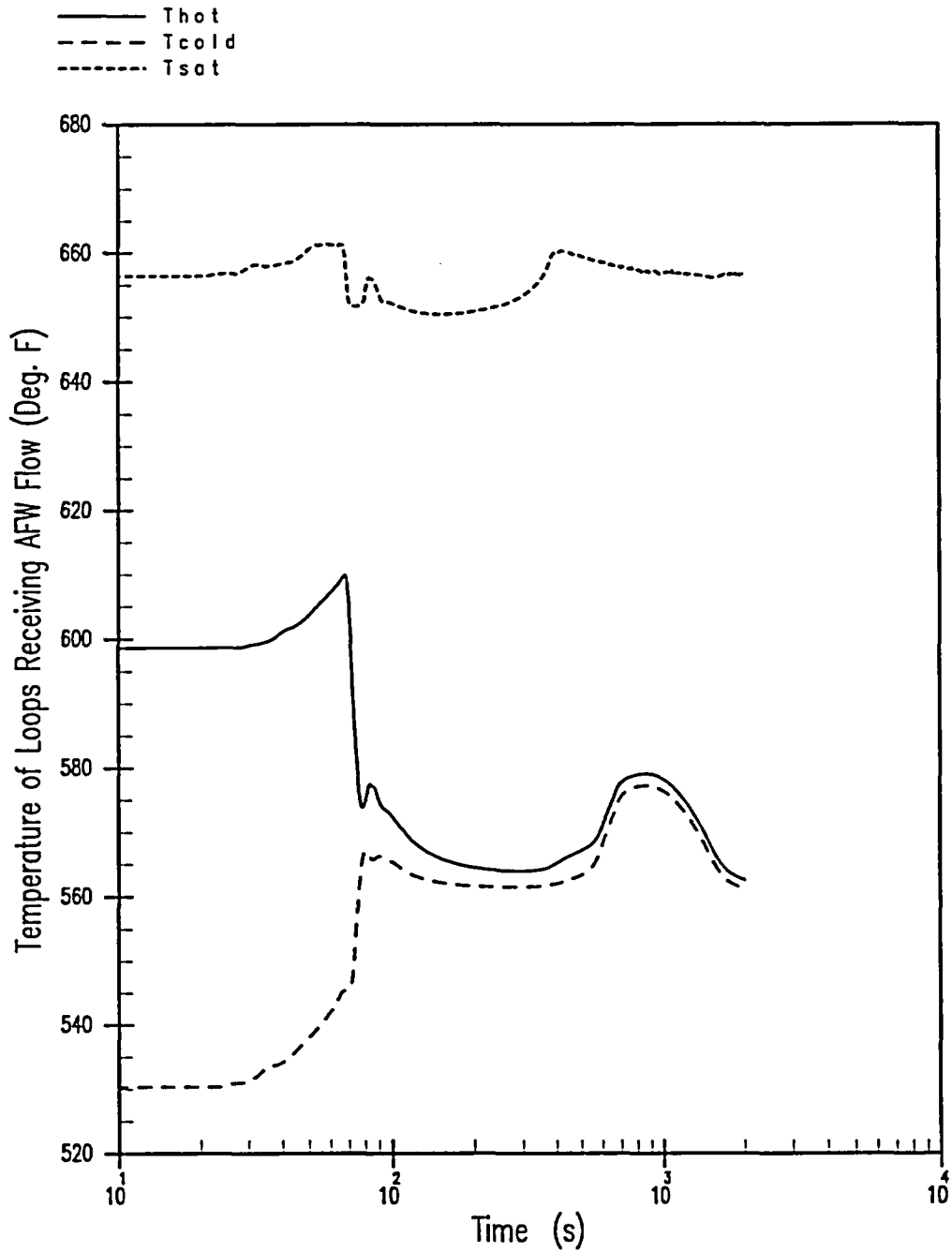


Figure 6.3-35
LONF RCS Temperatures for
Loops Receiving Automatic AFW Flow (Temperature vs. Time)

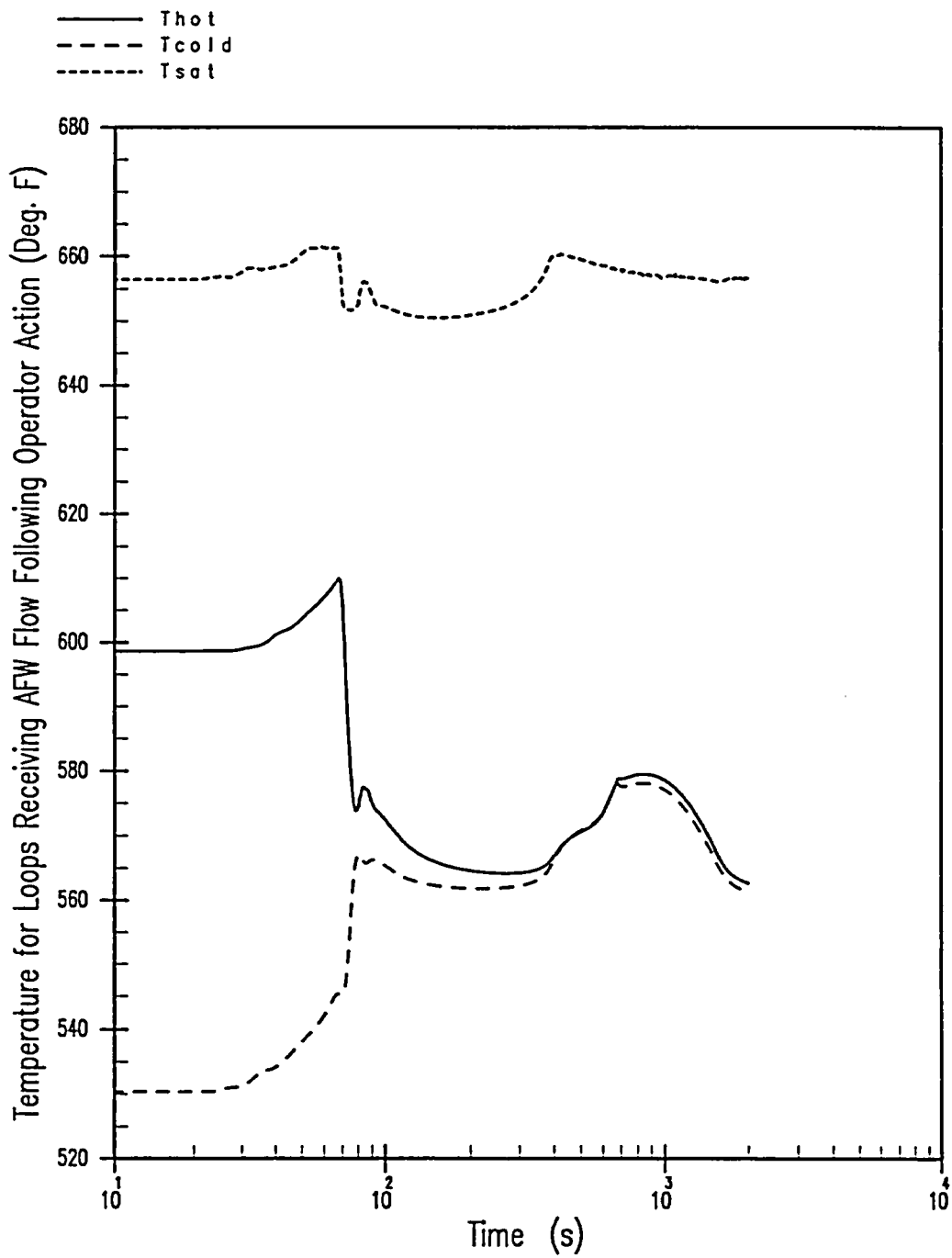


Figure 6.3-36
LONF RCS Temperatures for Loops
Receiving AFW Flow Following Operator Action (Temperature vs. Time)

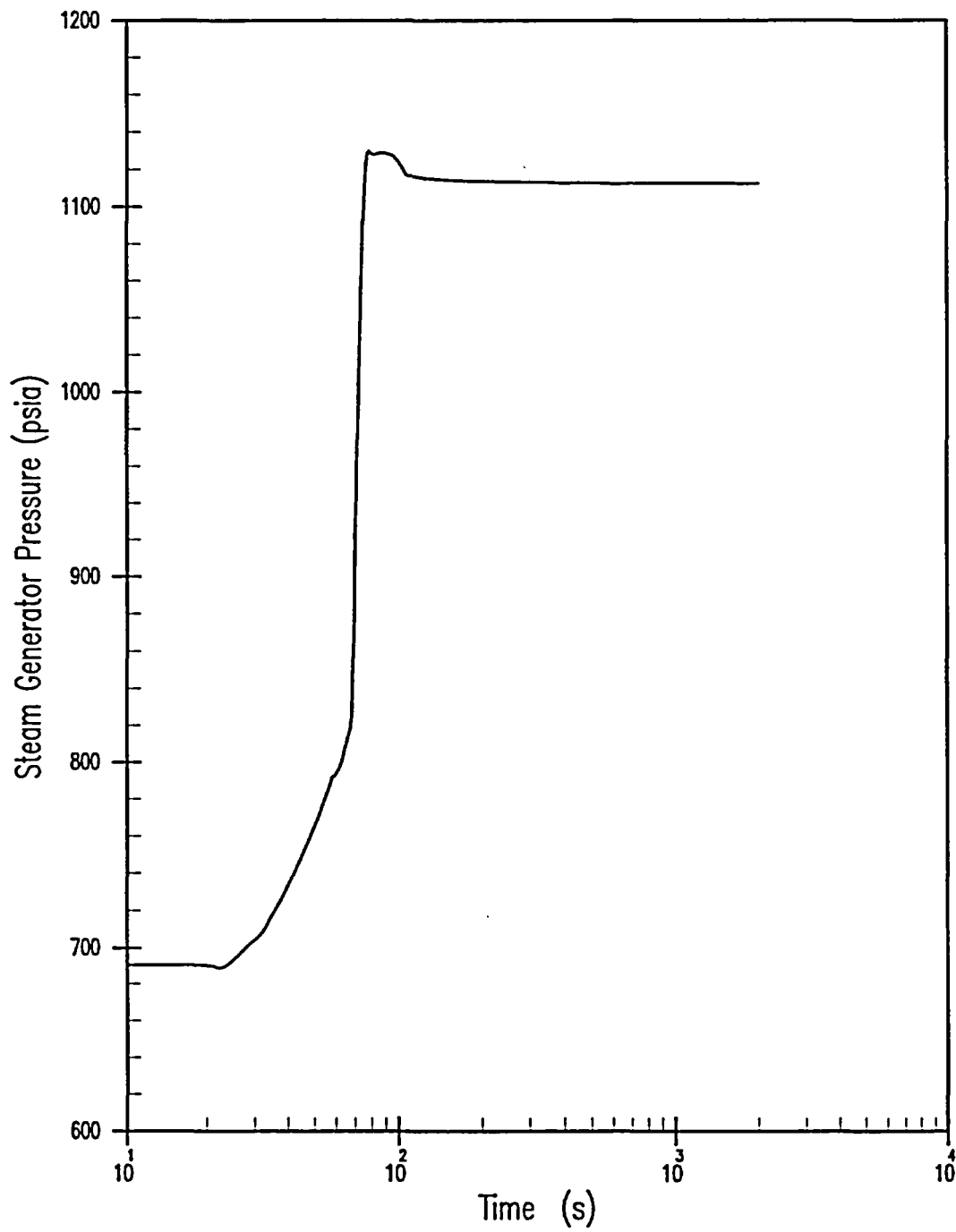


Figure 6.3-37
LONF (Steam Generator Pressure vs. Time)

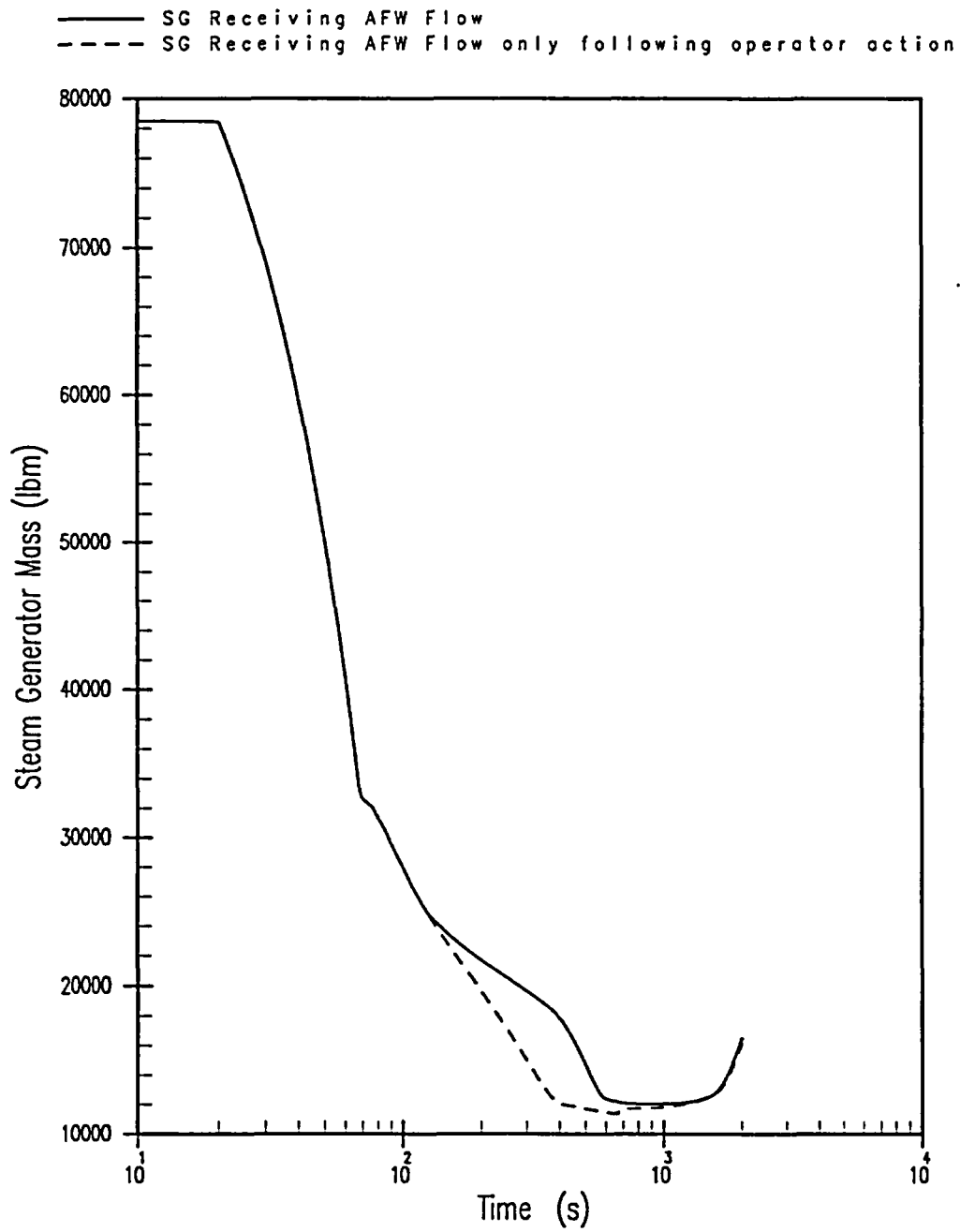


Figure 6.3-38
LONF (Steam Generator Mass vs. Time)

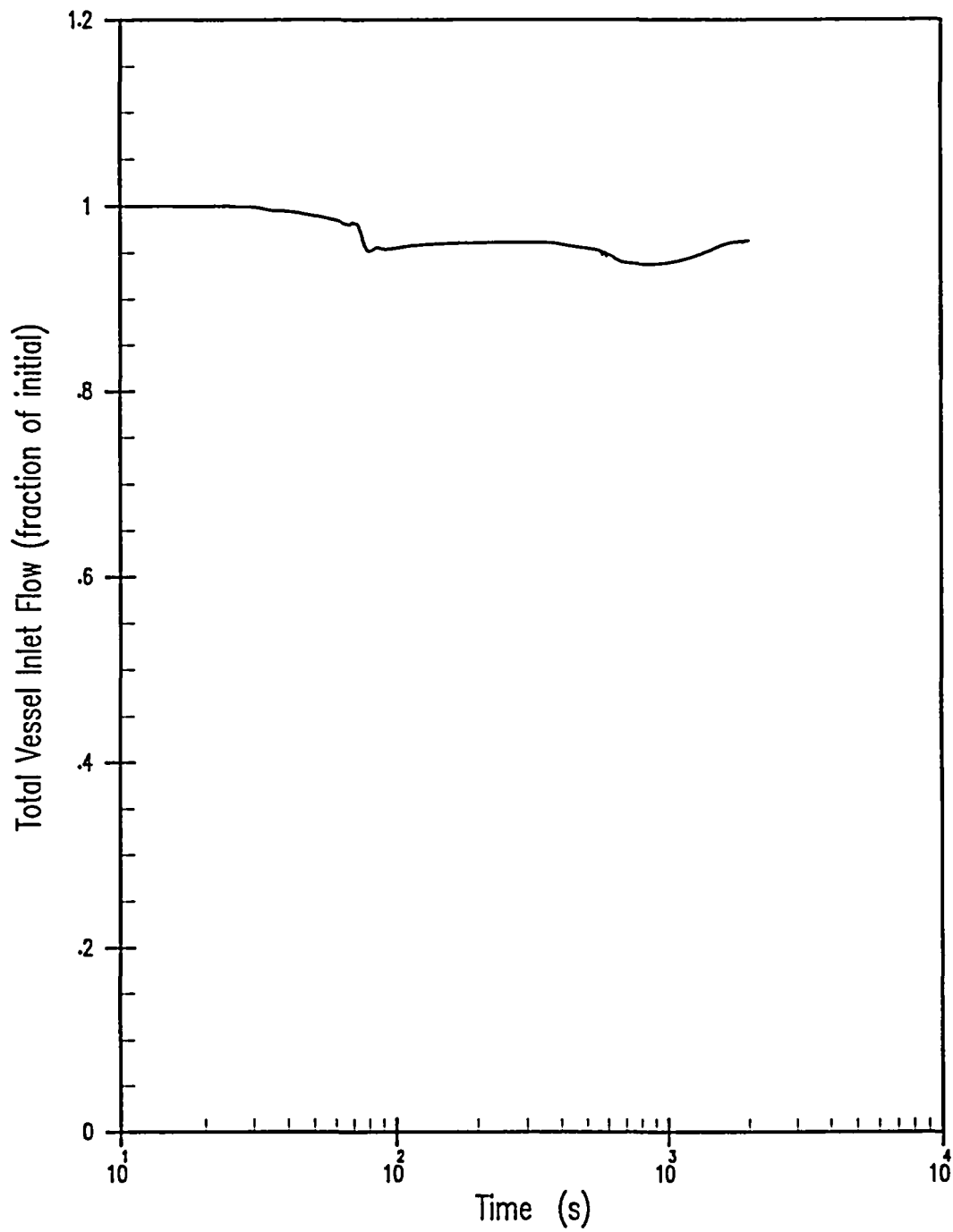


Figure 6.3-39
LONF (Total RCS Flow vs. Time)

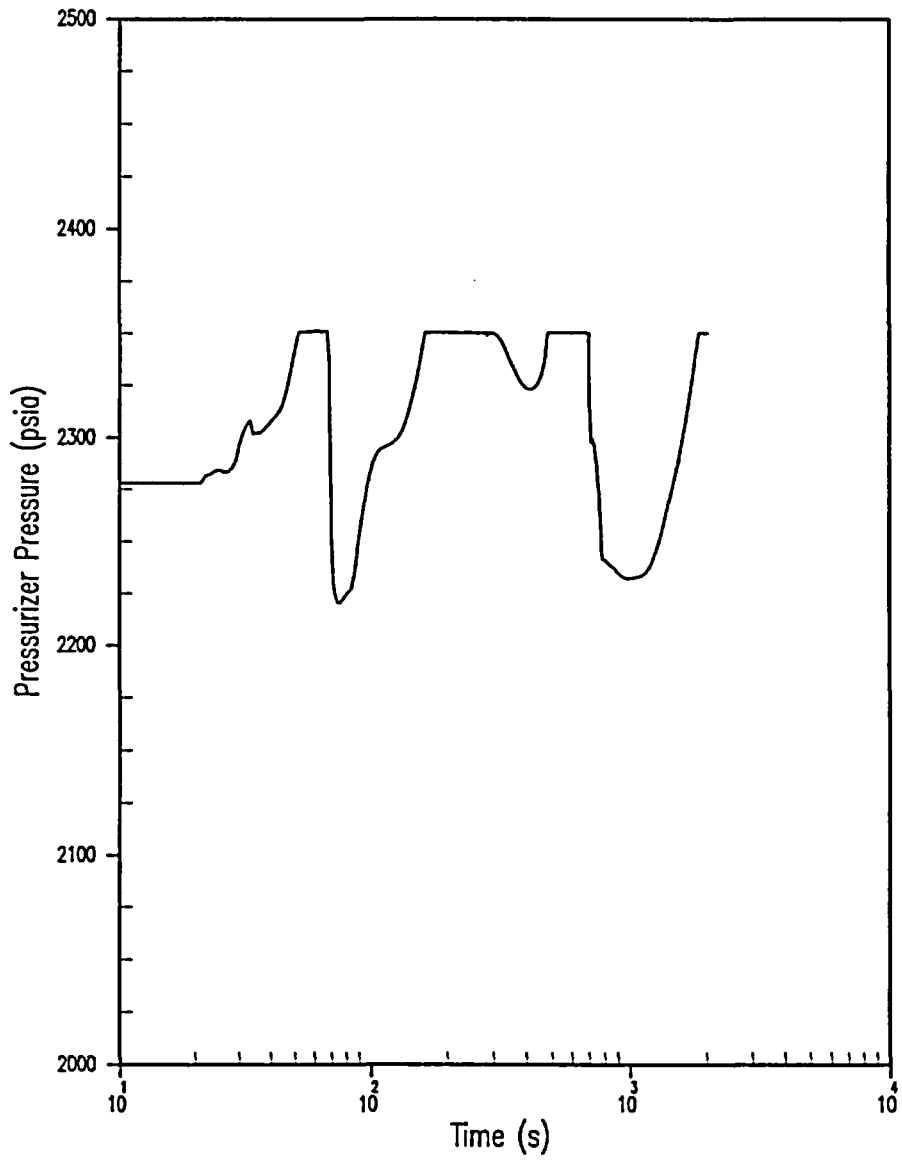


Figure 6.3-40
LOAC to the Plant Auxiliaries
(Pressurizer Pressure vs. Time)

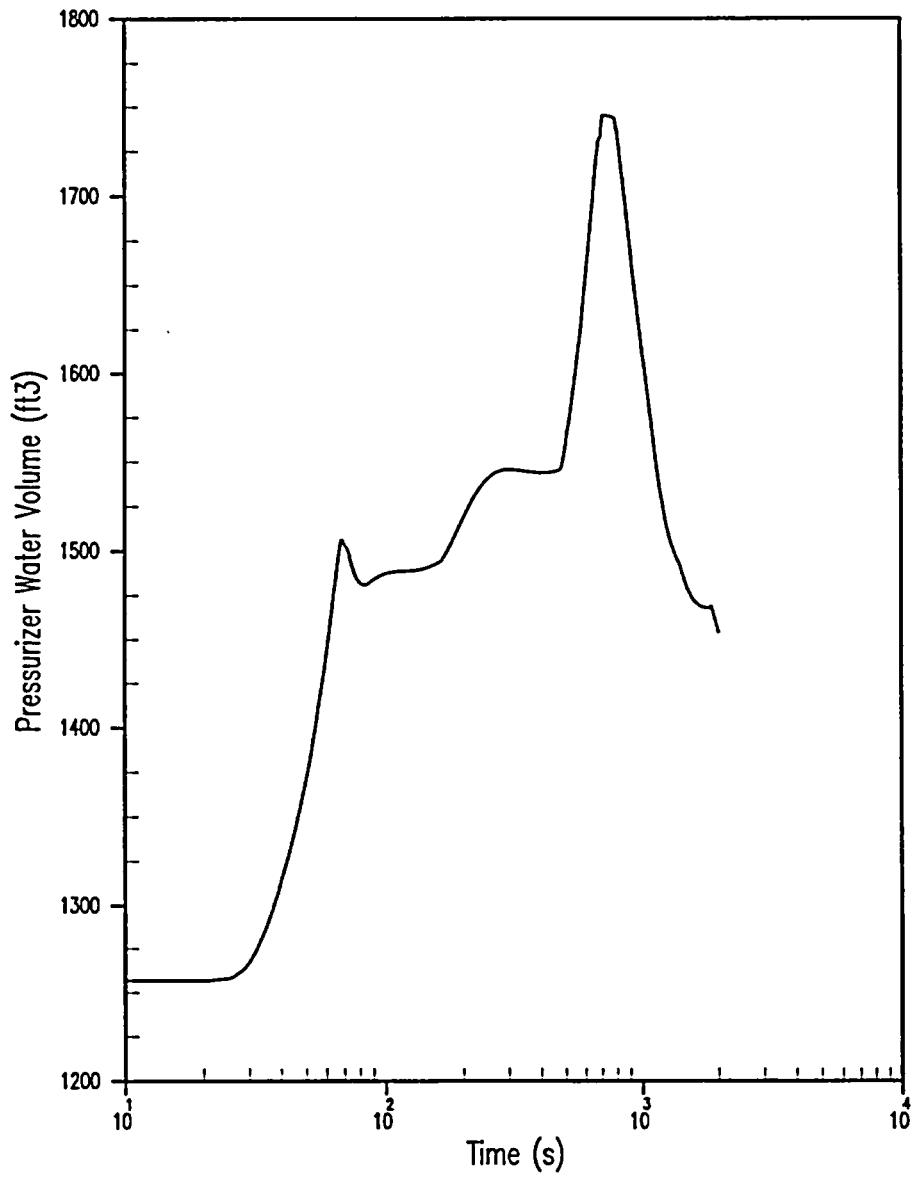


Figure 6.3-41
LOAC to the Plant Auxiliaries
(Pressurizer Water Volume vs. Time)

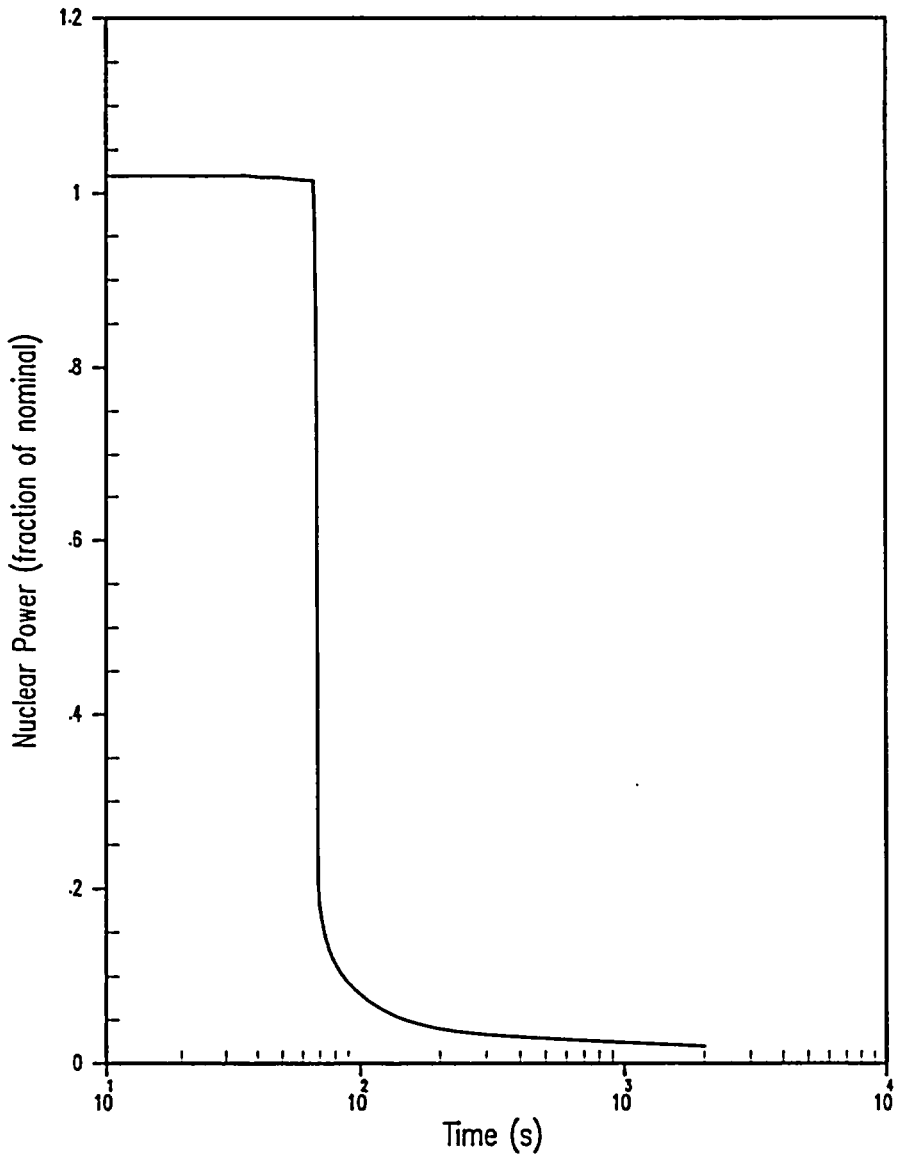


Figure 6.3-42
LOAC to the Plant Auxiliaries
(Nuclear Power vs. Time)

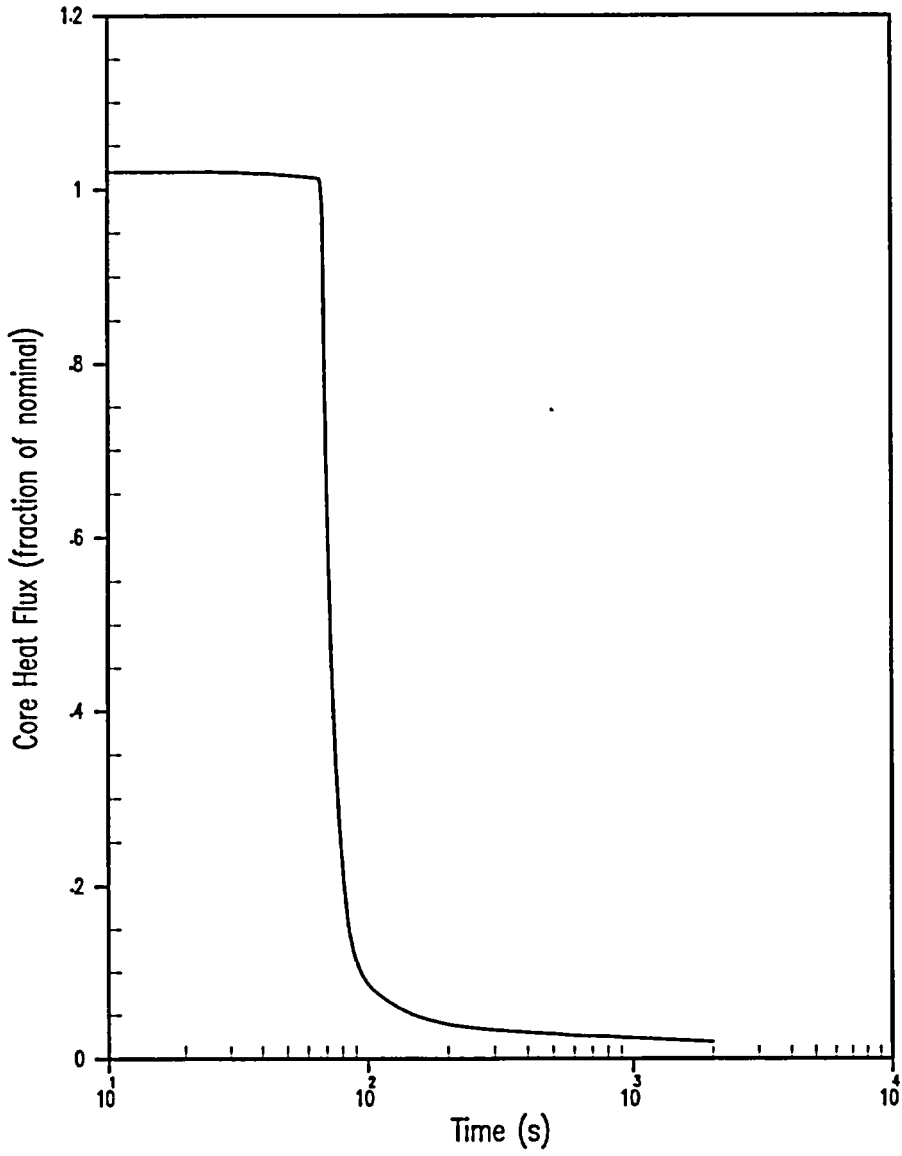


Figure 6.3-43
LOAC to the Plant Auxiliaries
(Core Heat Flux vs. Time)

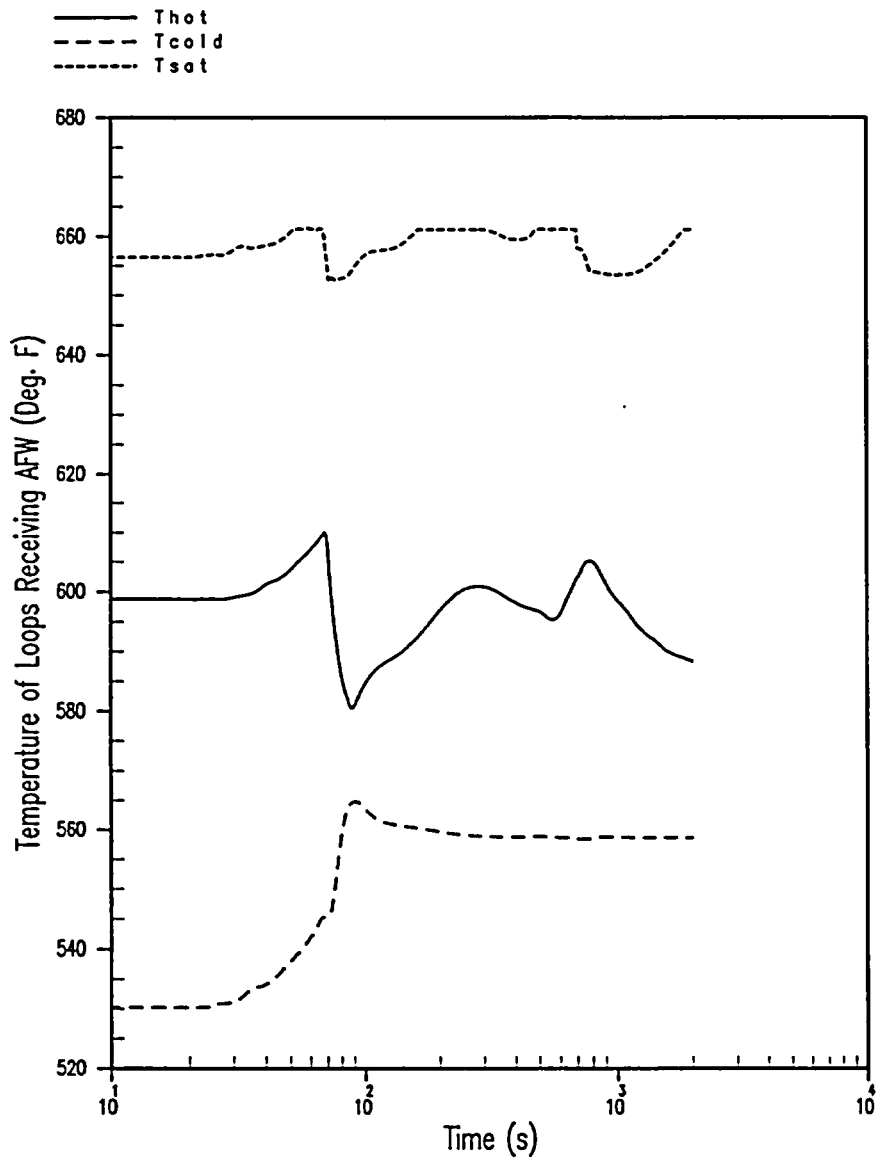


Figure 6.3-44
LOAC to the Plant Auxiliaries RCS Temperatures for
Loops Receiving Automatic AFW Flow (Temperature vs. Time)

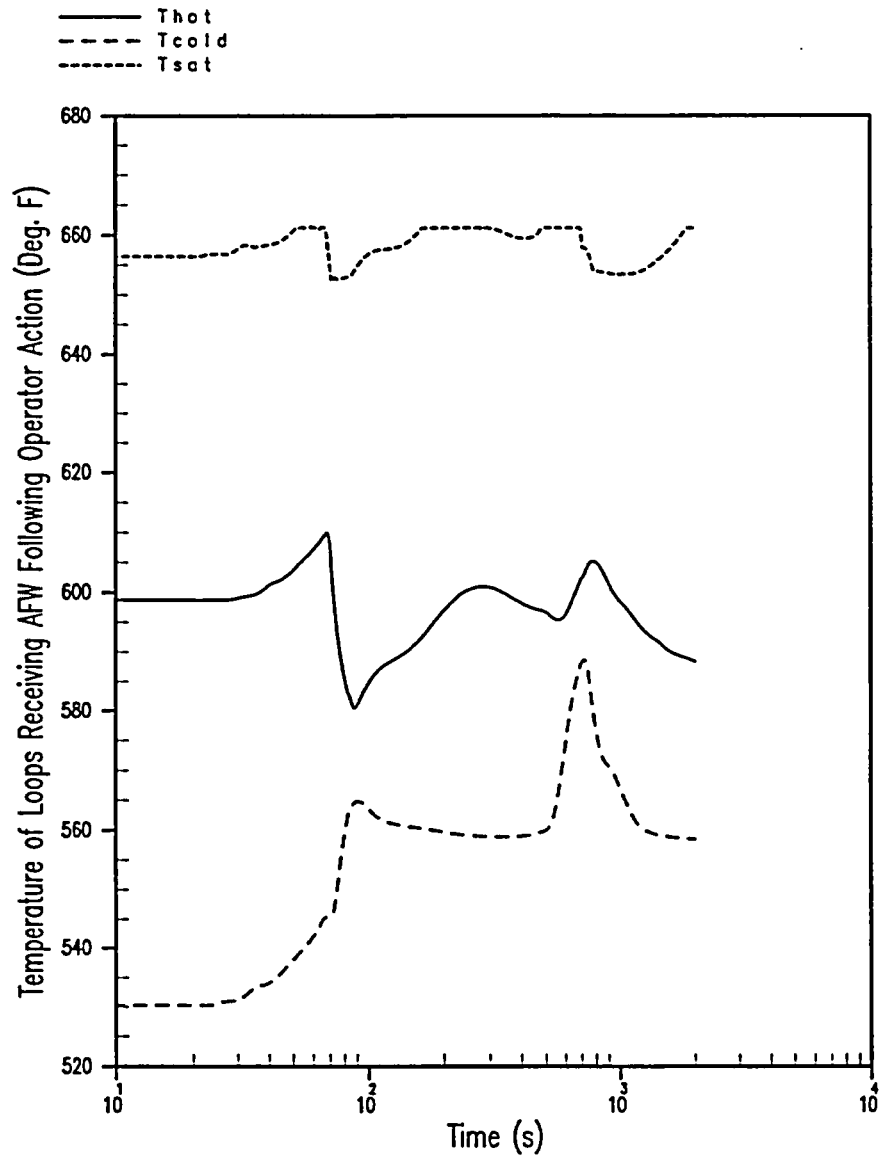


Figure 6.3-45
LOAC to the Plant Auxiliaries RCS Temperatures for Loops
Receiving AFW Flow Following Operator Action (Temperature vs. Time)

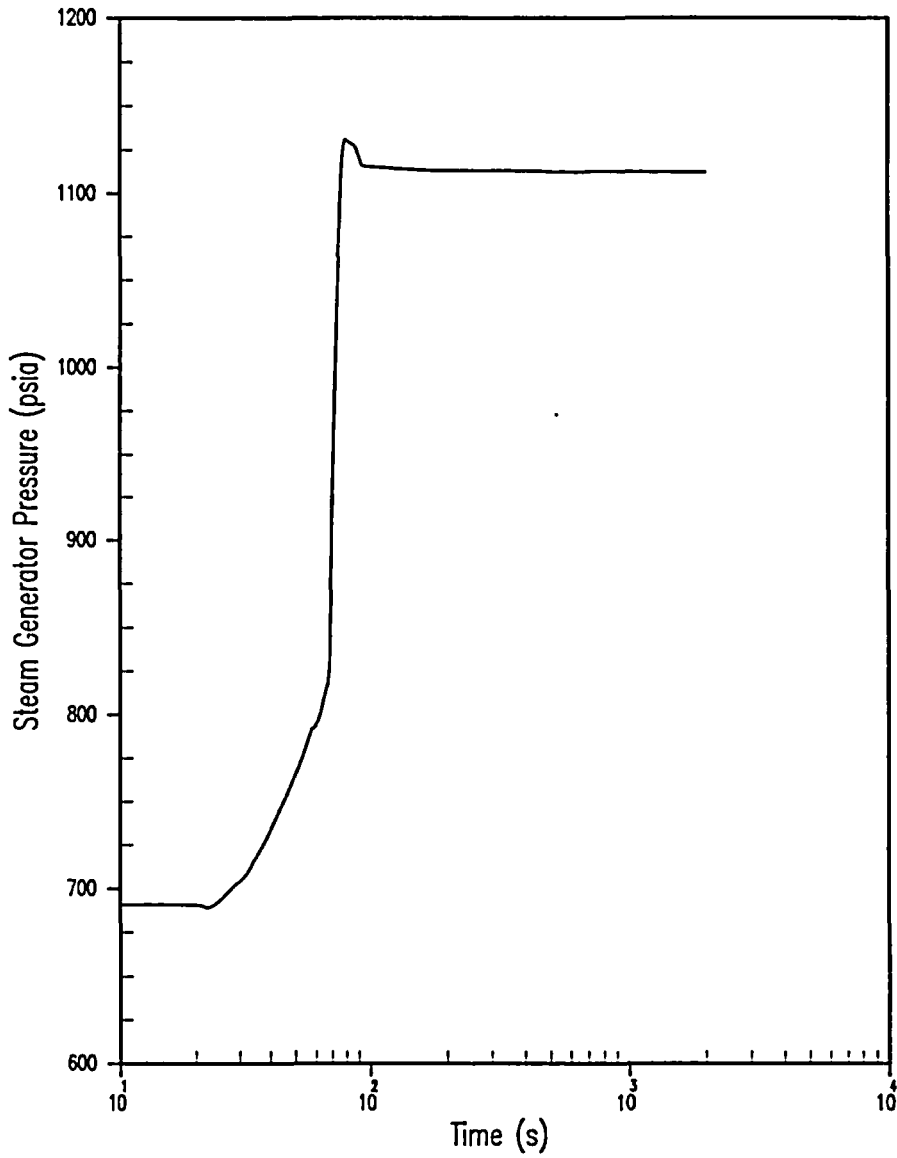


Figure 6.3-46
LOAC to the Plant Auxiliaries
(Steam Generator Pressure vs. Time)

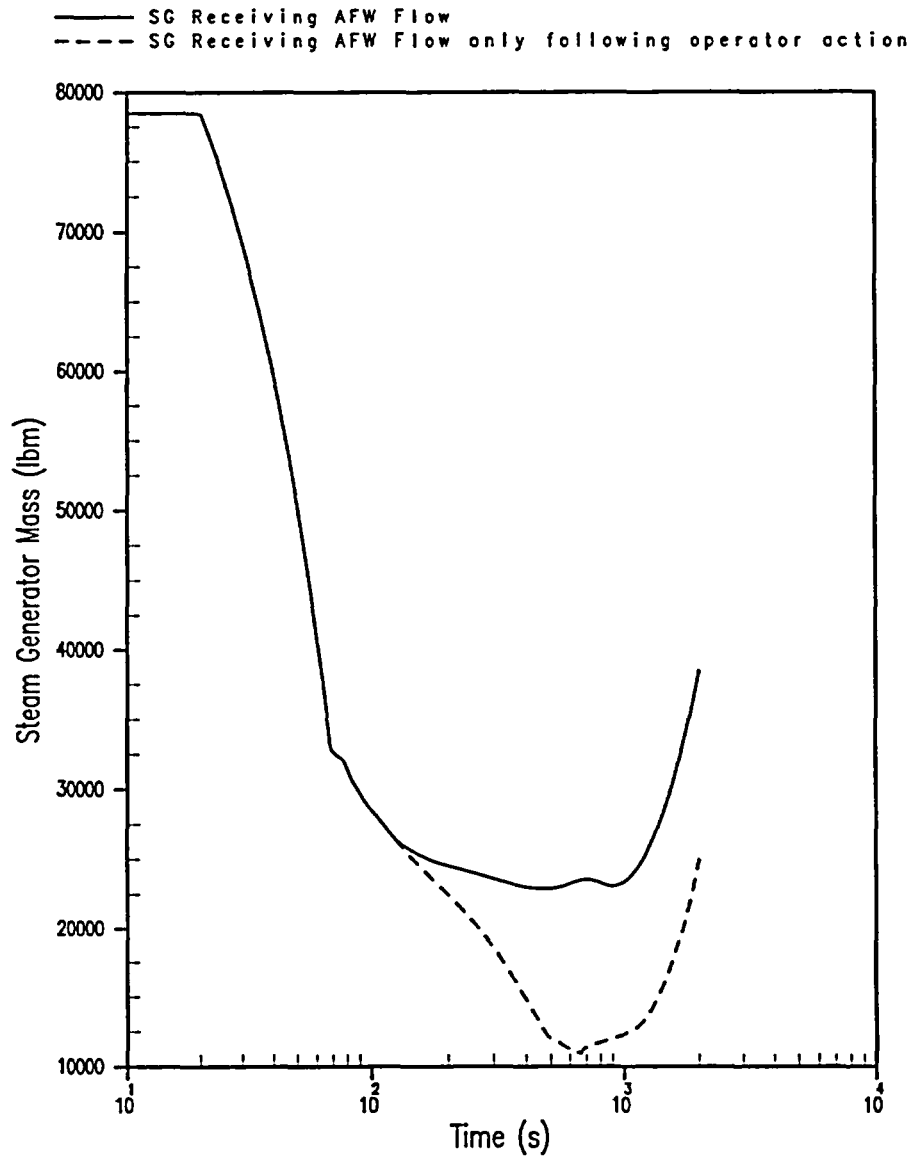


Figure 6.3-47
LOAC to the Plant Auxiliaries
(Steam Generator Mass vs. Time)

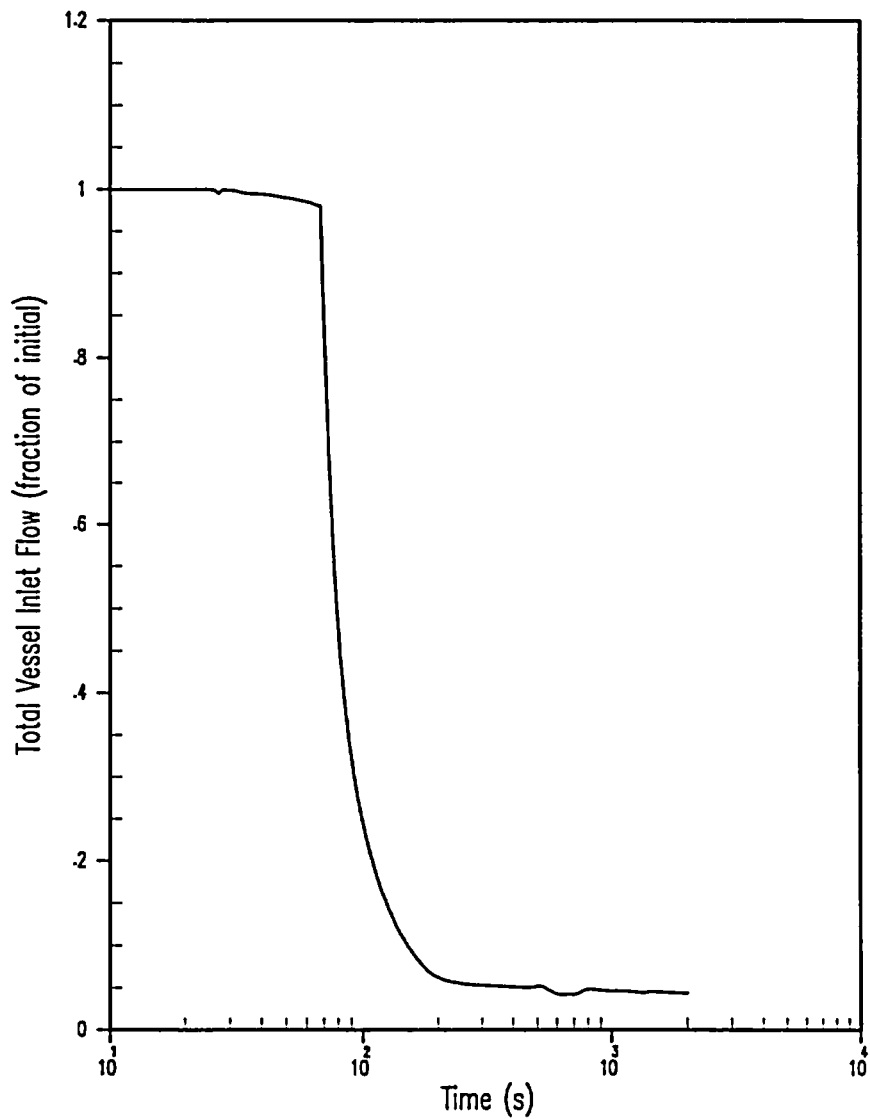


Figure 6.3-48
LOAC to the Plant Auxiliaries
(Total RCS Flow vs. Time)

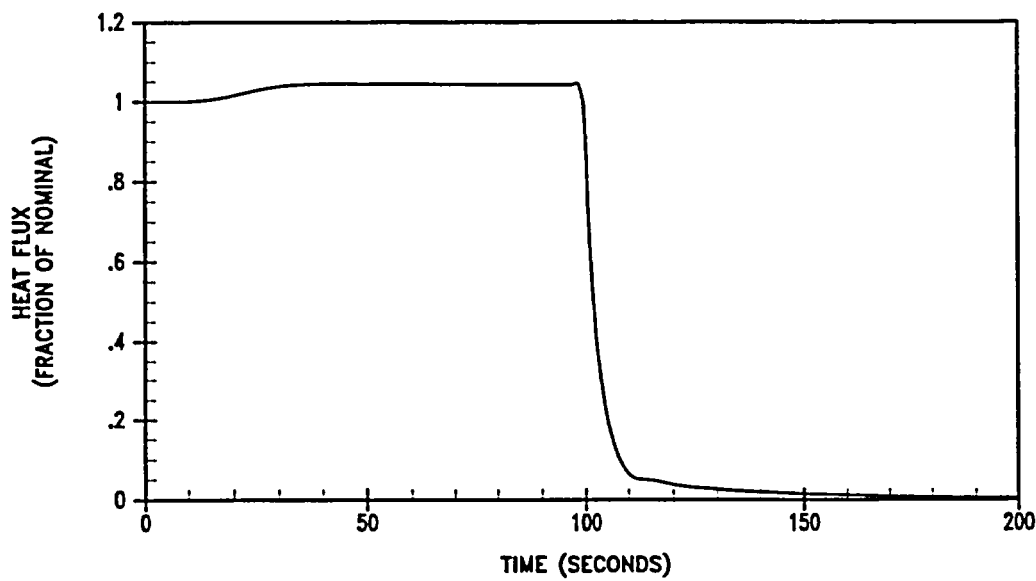
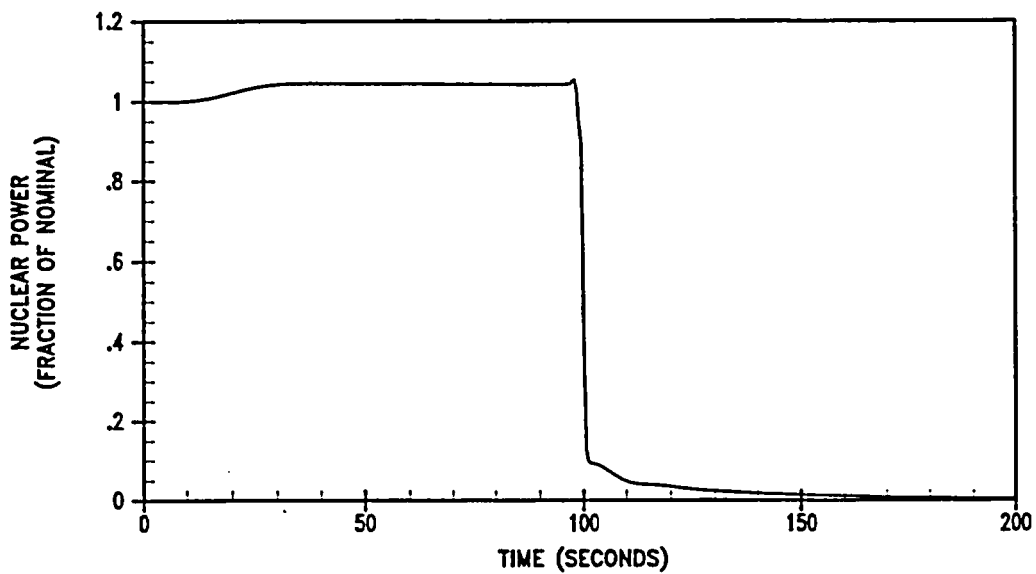


Figure 6.3-49
Feedwater System Malfunction at Full Power
(Nuclear Power and Core Heat Flux vs. Time)

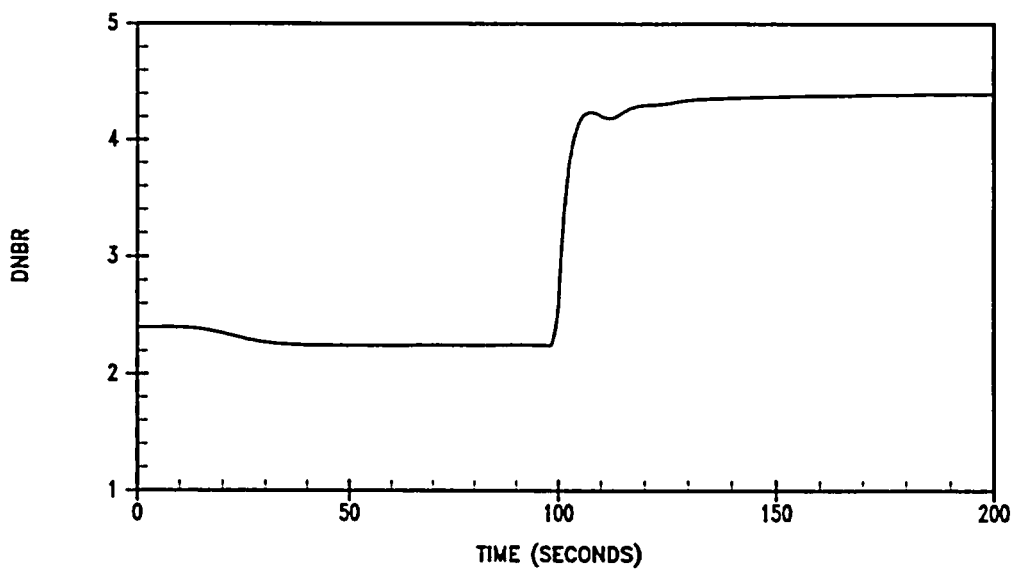
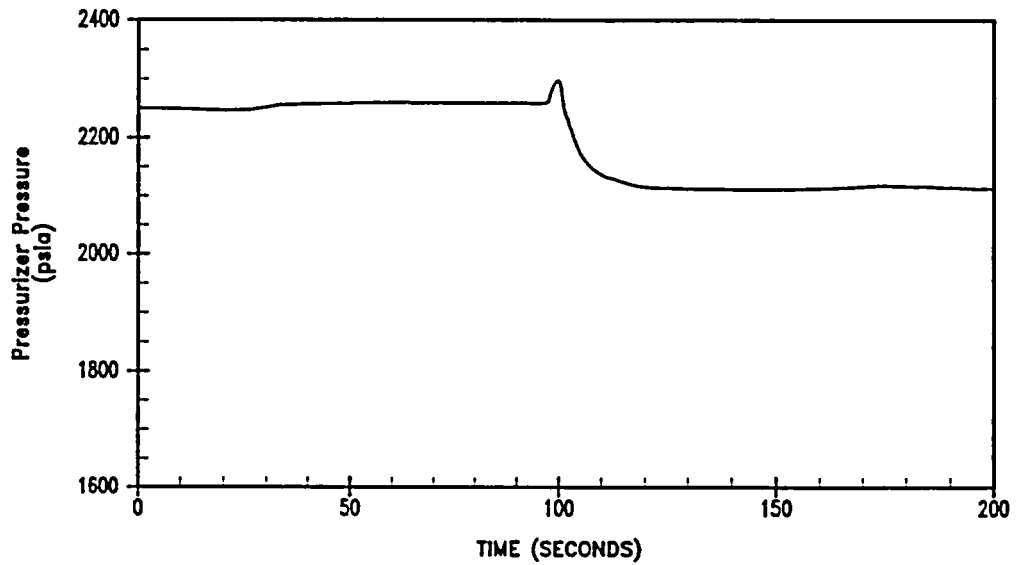


Figure 6.3-50
Feedwater System Malfunction at Full Power
(Pressurizer Pressure and DNBR vs. Time)

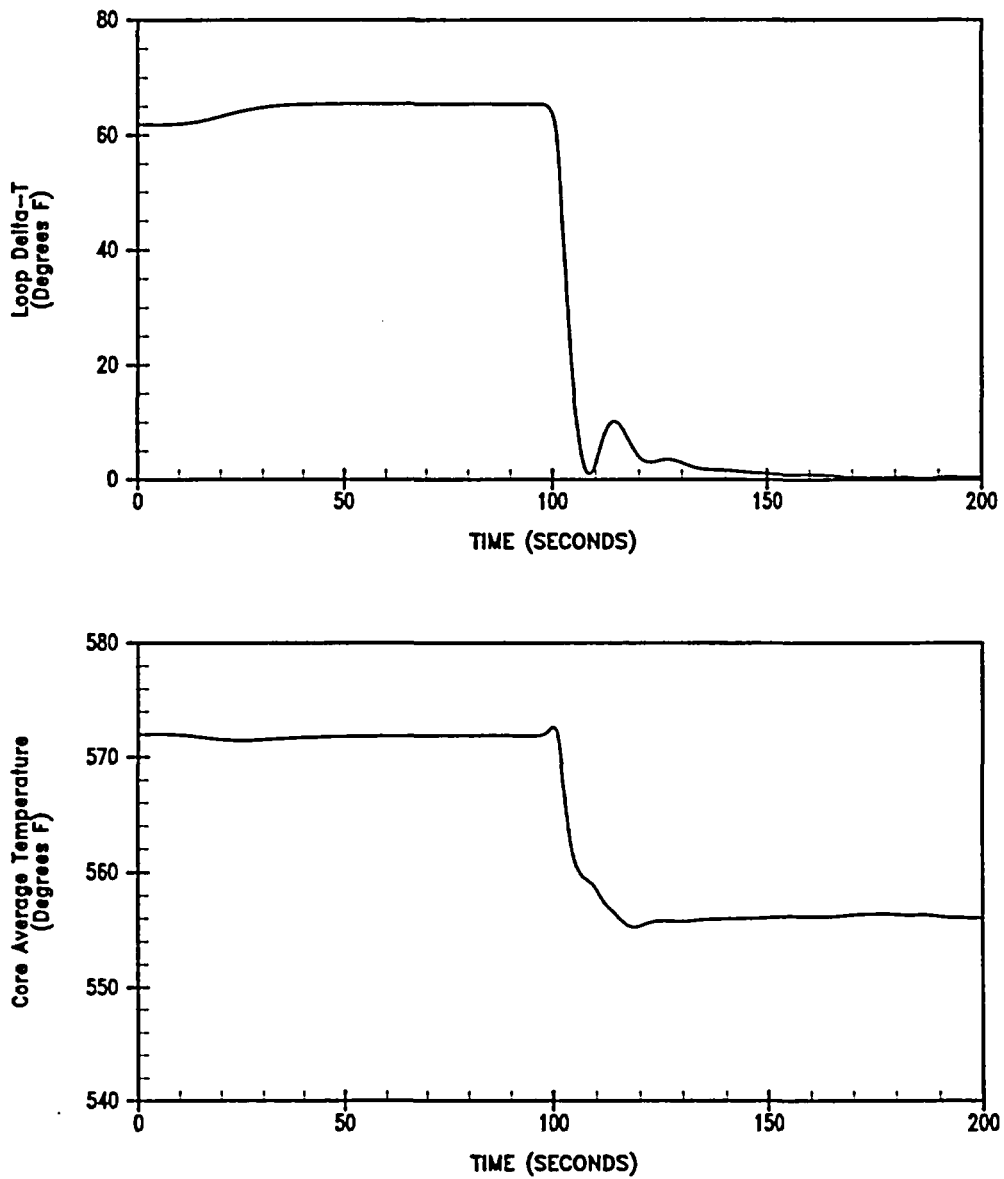


Figure 6.3-51
Feedwater System Malfunction at Full Power
(Loop Delta-T and Core Average Temperature vs. Time)

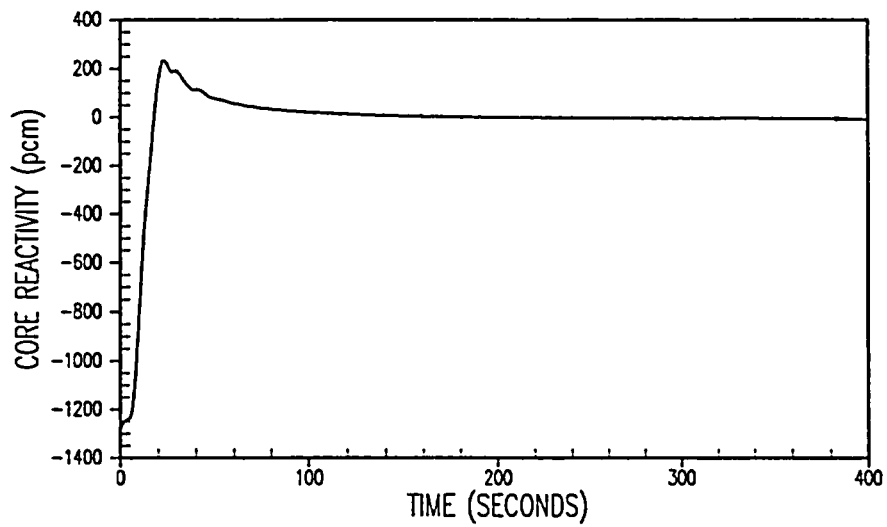
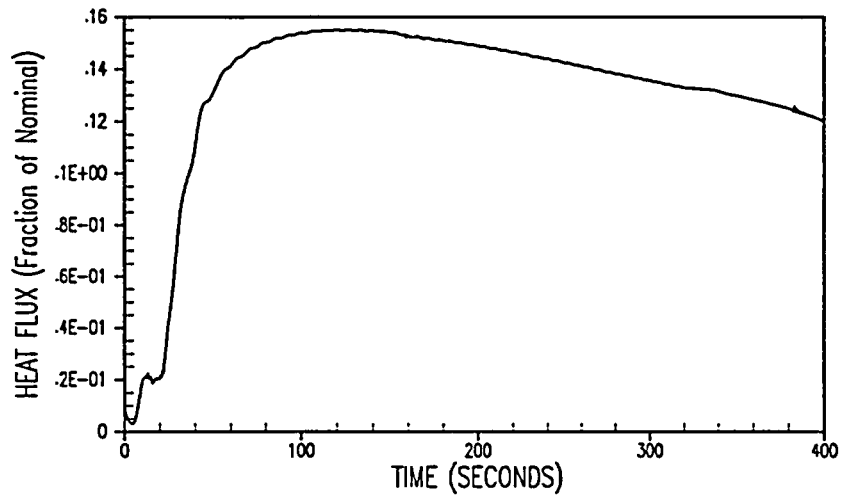


Figure 6.3-52
1.4 ft² Steamline Break, Offsite Power Available
(Core Heat Flux and Core Reactivity vs. Time)

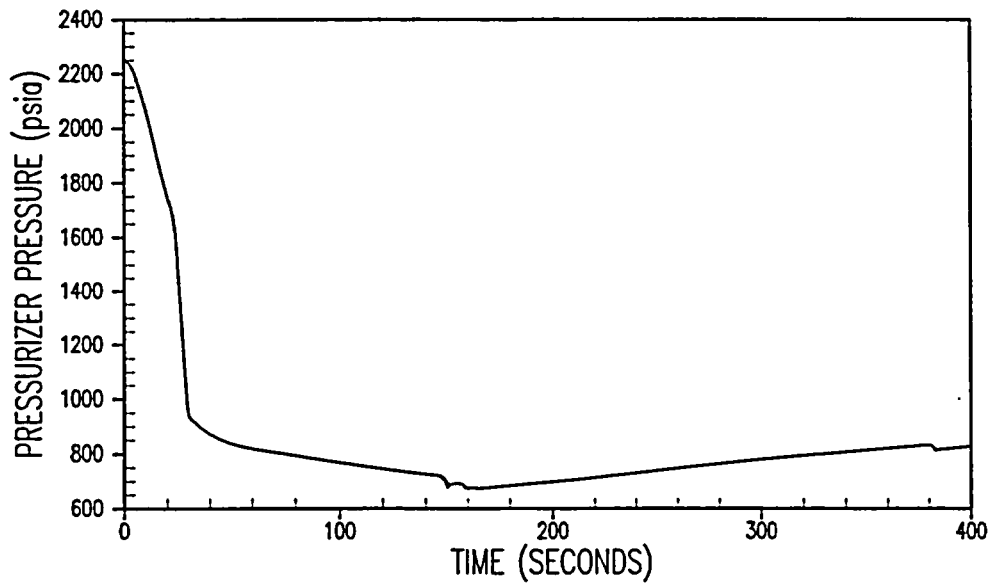
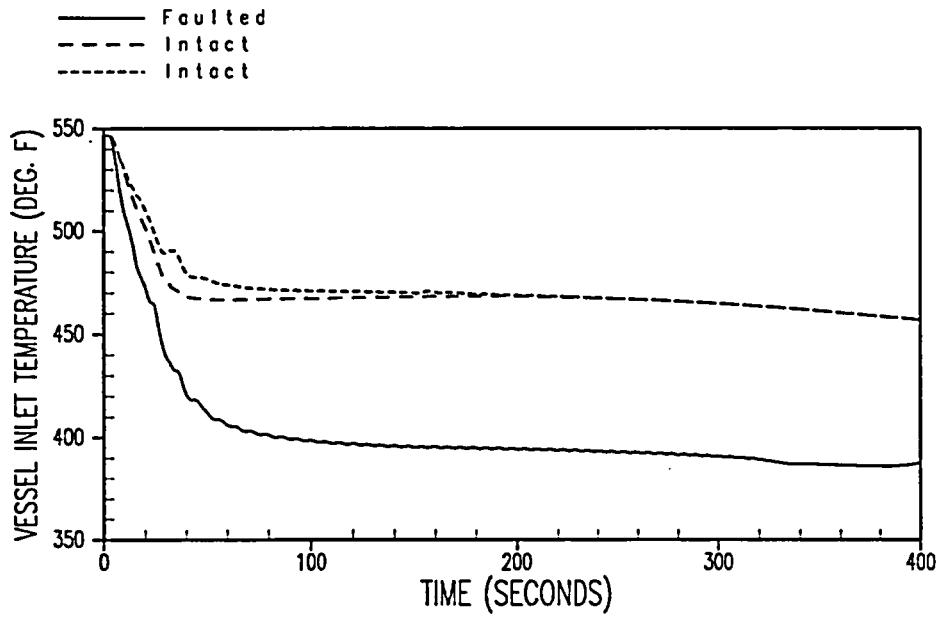


Figure 6.3-53
1.4 ft² Steamline Break, Offsite Power Available
(Reactor Vessel Inlet Temperature and Pressurizer Pressure vs. Time)

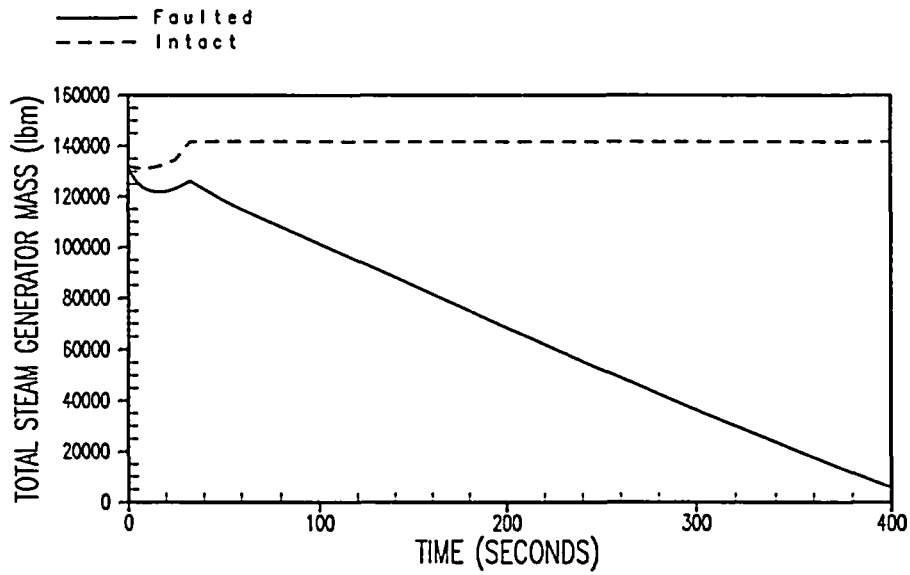
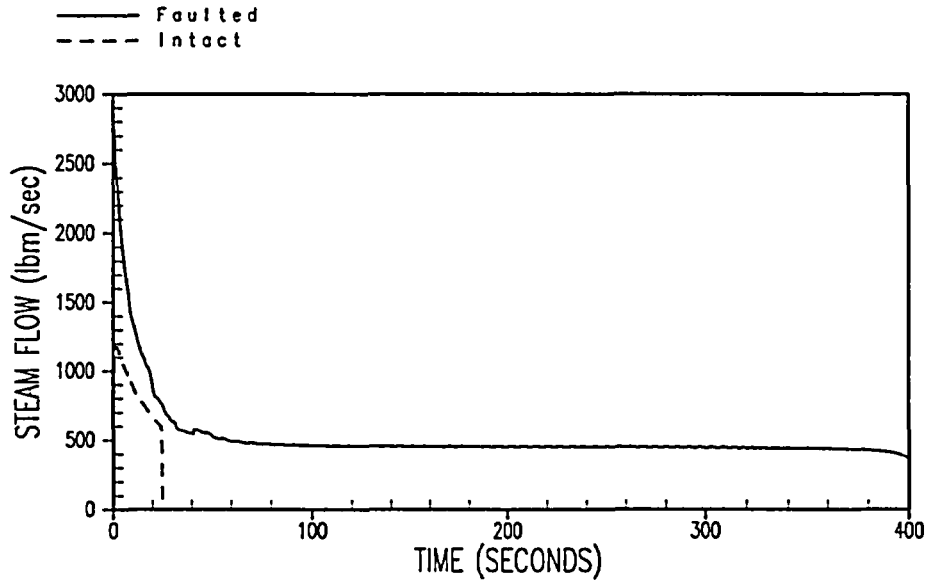


Figure 6.3-54
1.4 ft² Steamline Break, Offsite Power Available
(Steam Flow and Steam Generator Mass vs. Time)

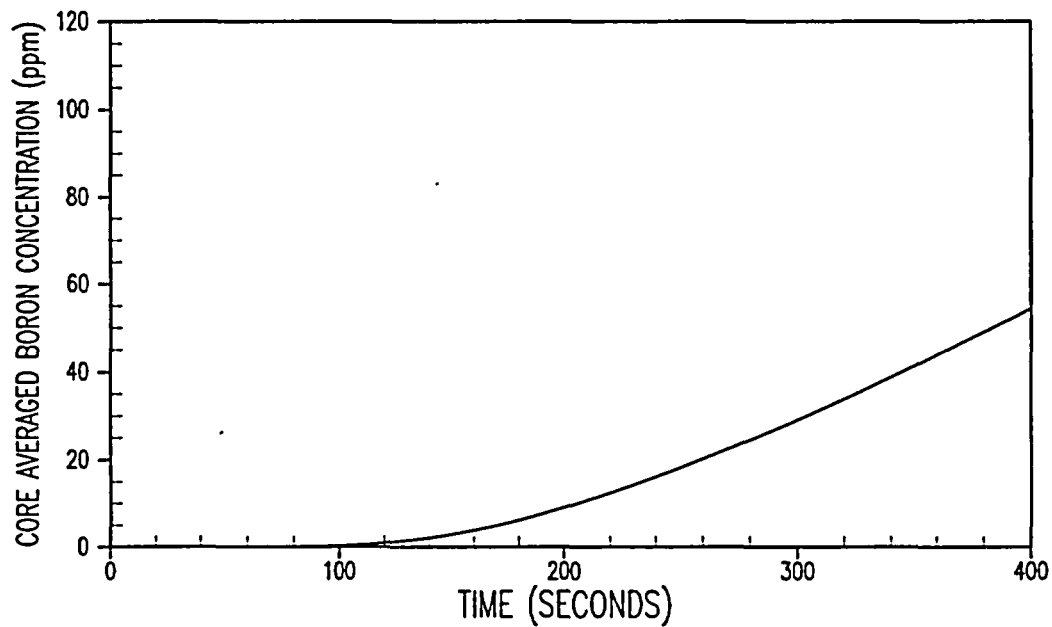


Figure 6.3-55
1.4 ft² Steamline Break, Offsite Power Available
(Core Averaged Boron Concentration vs. Time)

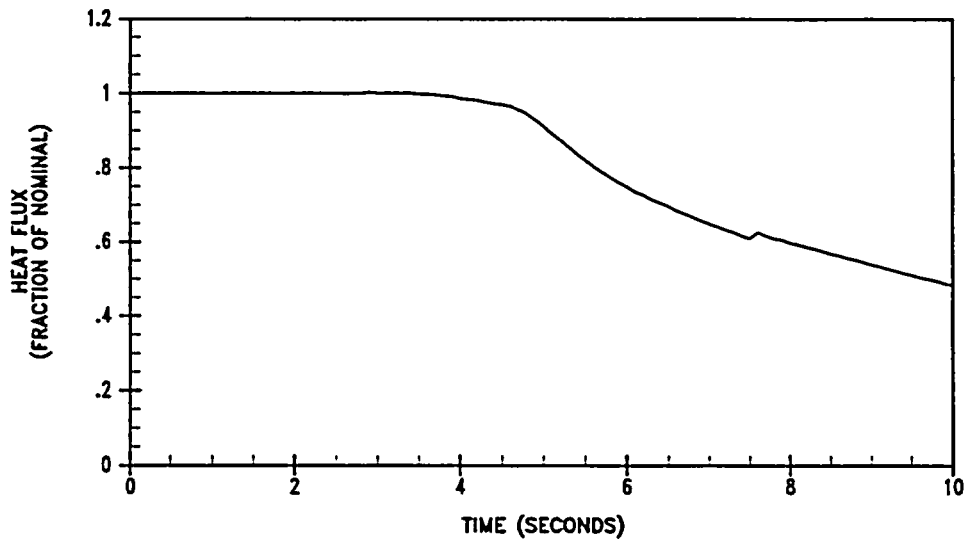
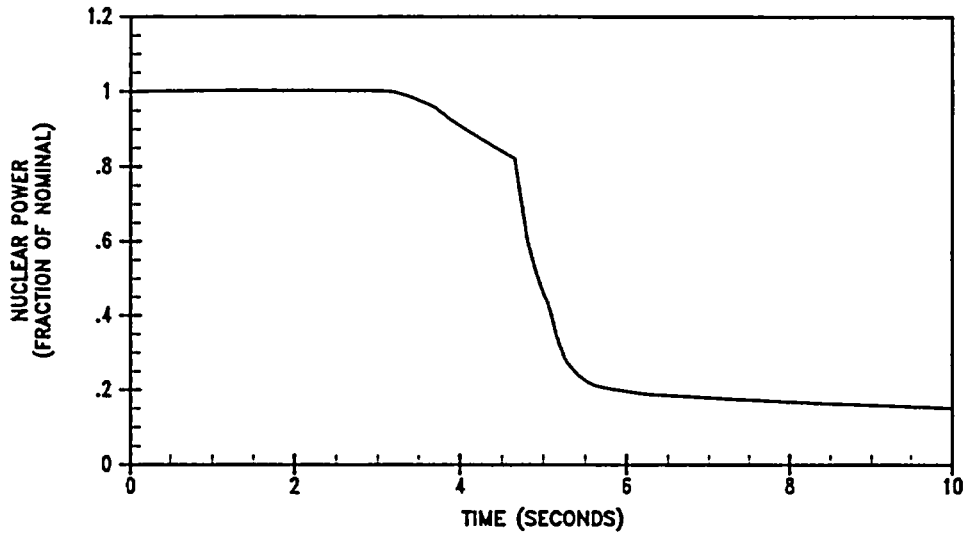


Figure 6.3-56
Partial Loss of Forced Reactor Coolant Flow
(Nuclear Power and Heat Flux vs. Time)

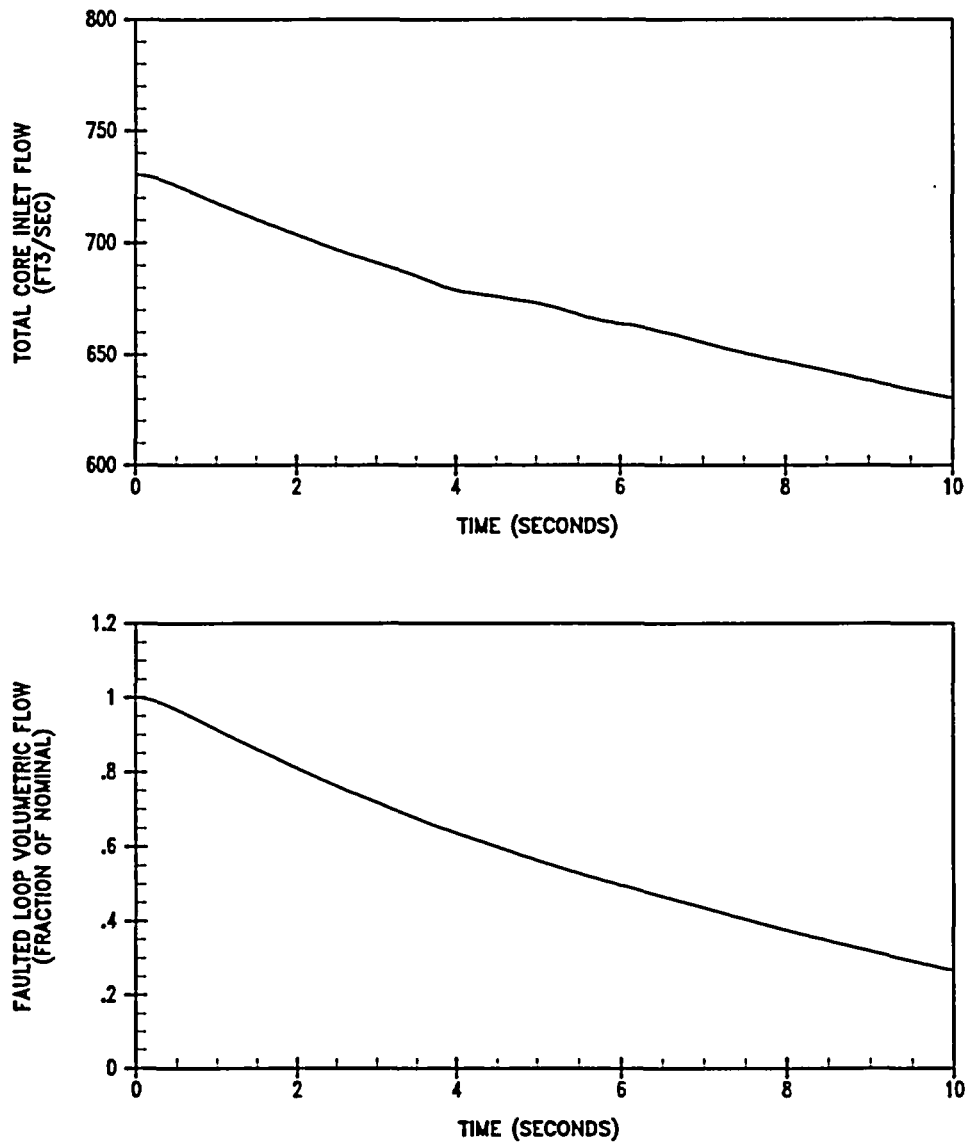


Figure 6.3-57
Partial Loss of Forced Reactor Coolant Flow
(Total Core Flow and Faulted Loop Flow vs. Time)

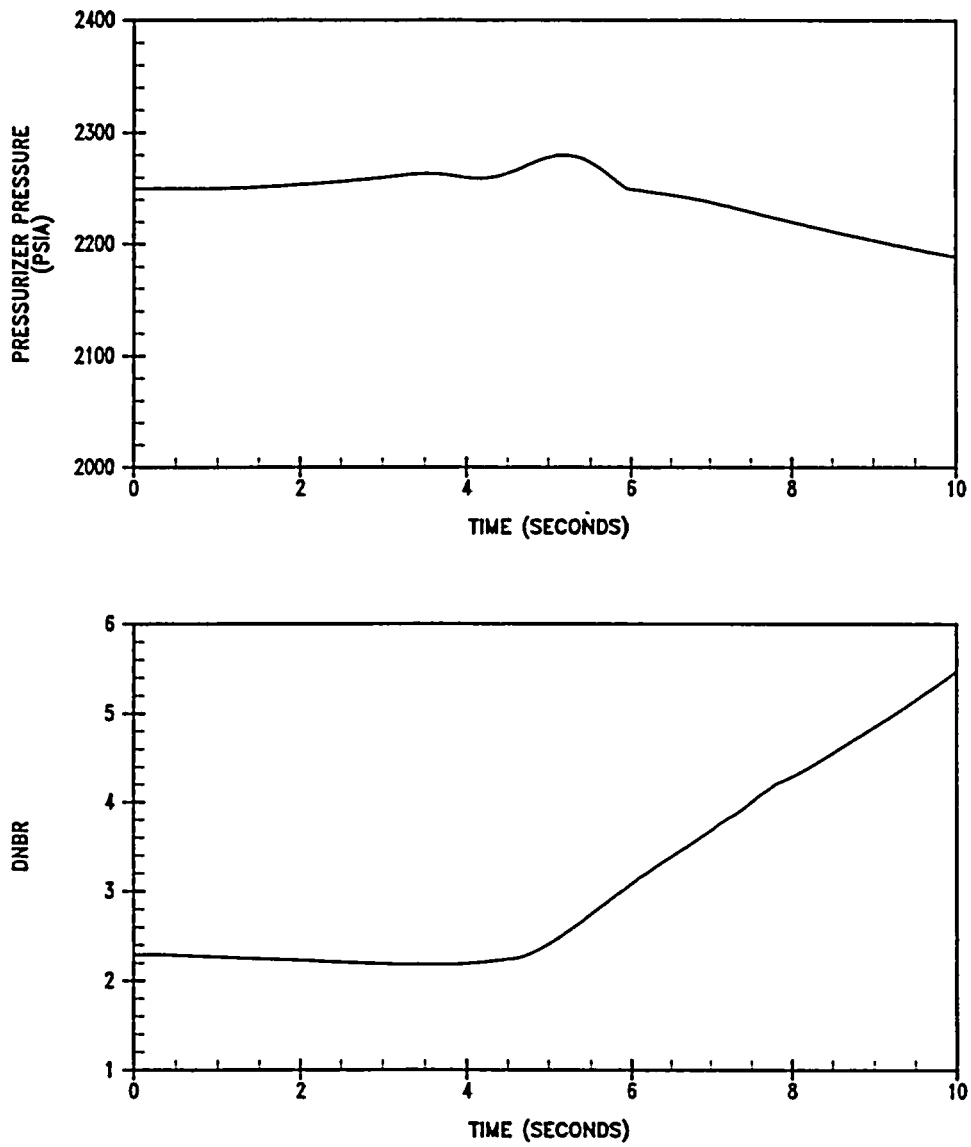


Figure 6.3-58
Partial Loss of Forced Reactor Coolant Flow
(Pressurizer Pressure and DNBR vs. Time)

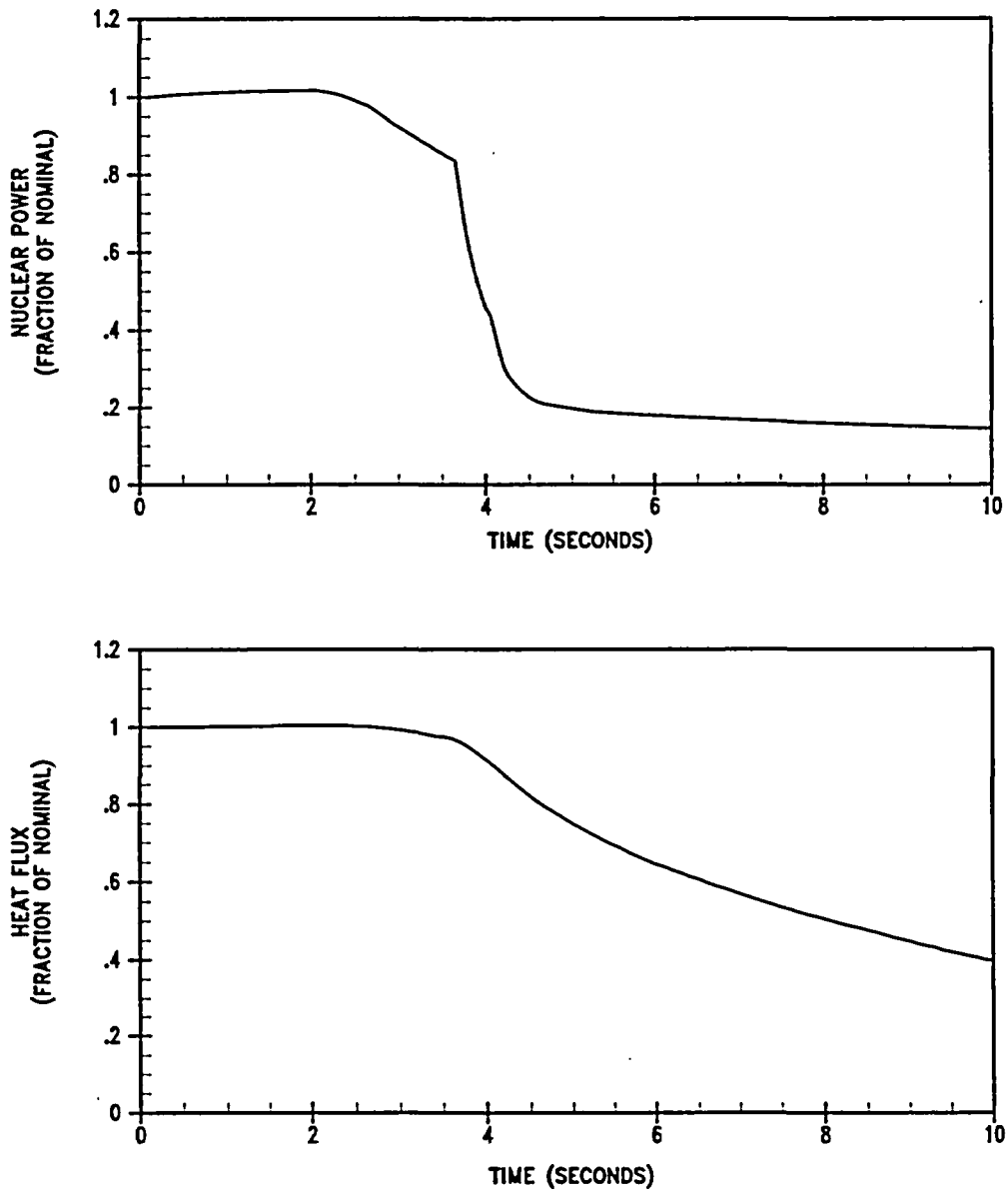


Figure 6.3-59
Complete Loss of Forced Reactor Coolant Flow – Frequency Decay
(Nuclear Power and Heat Flux vs. Time)

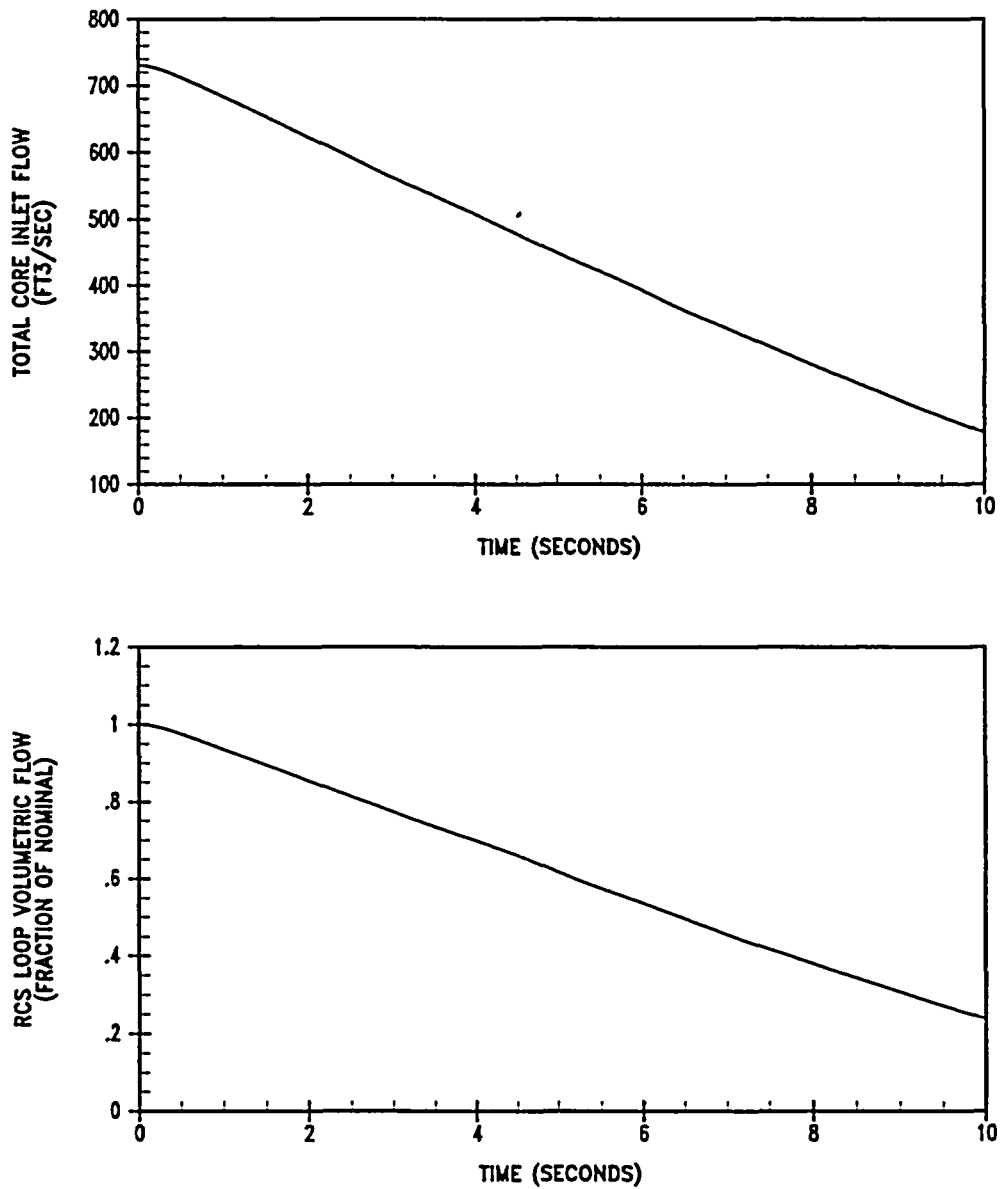


Figure 6.3-60
Complete Loss of Forced Reactor Coolant Flow – Frequency Decay
(Total Core Flow and RCS Loop Flow vs. Time)

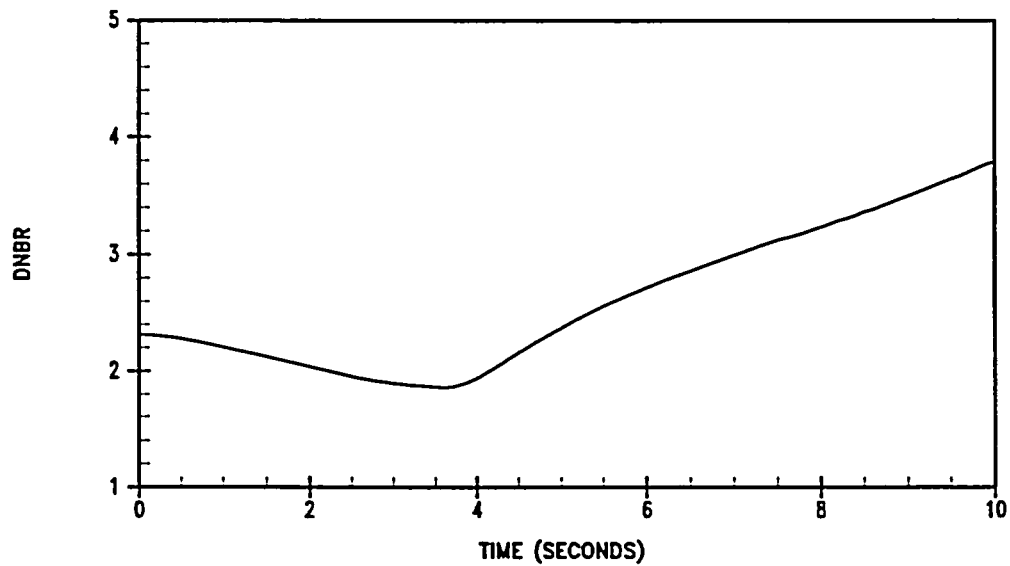
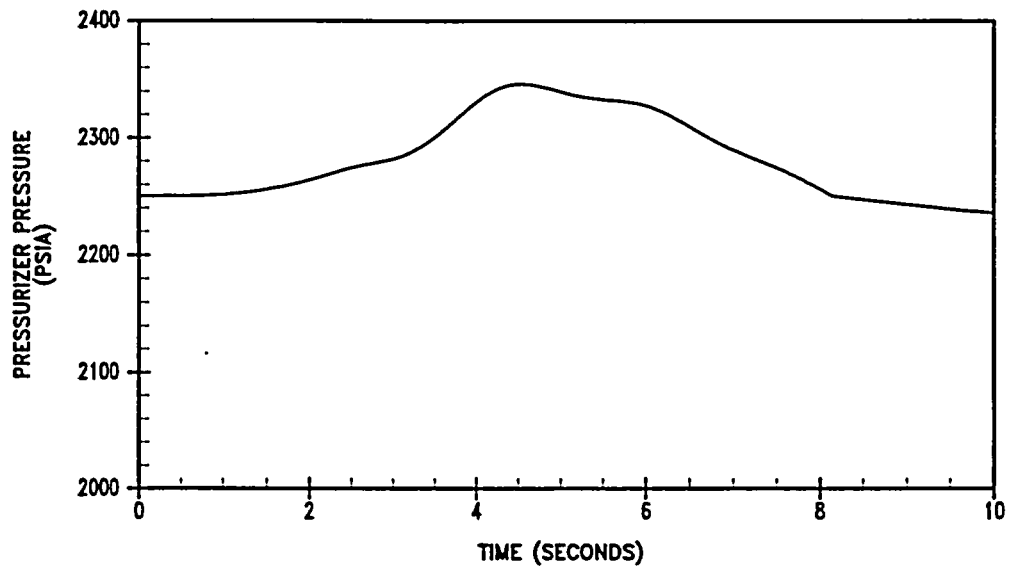


Figure 6.3-61
Complete Loss of Forced Reactor Coolant Flow – Frequency Decay
(Pressurizer Pressure and DNBR vs. Time)

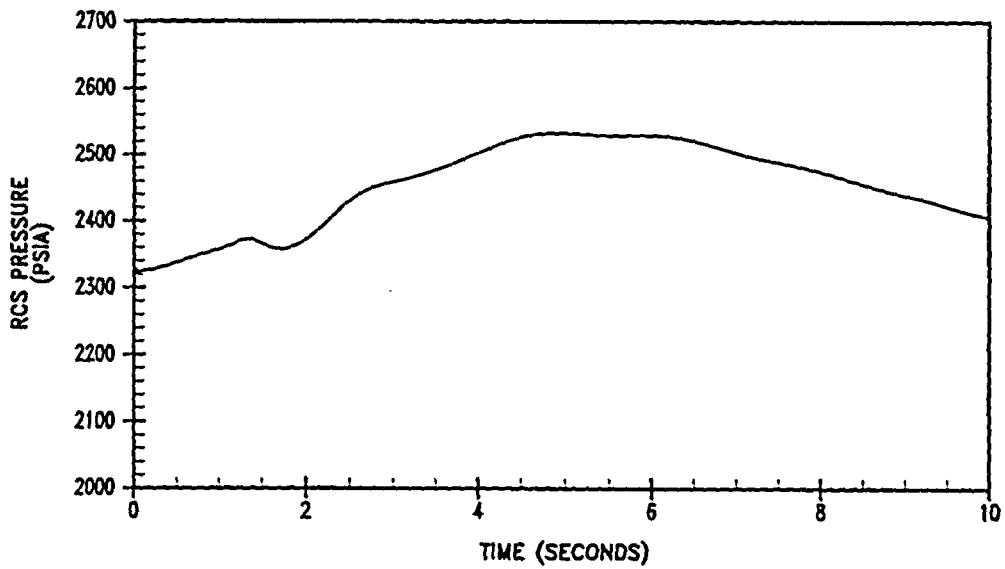
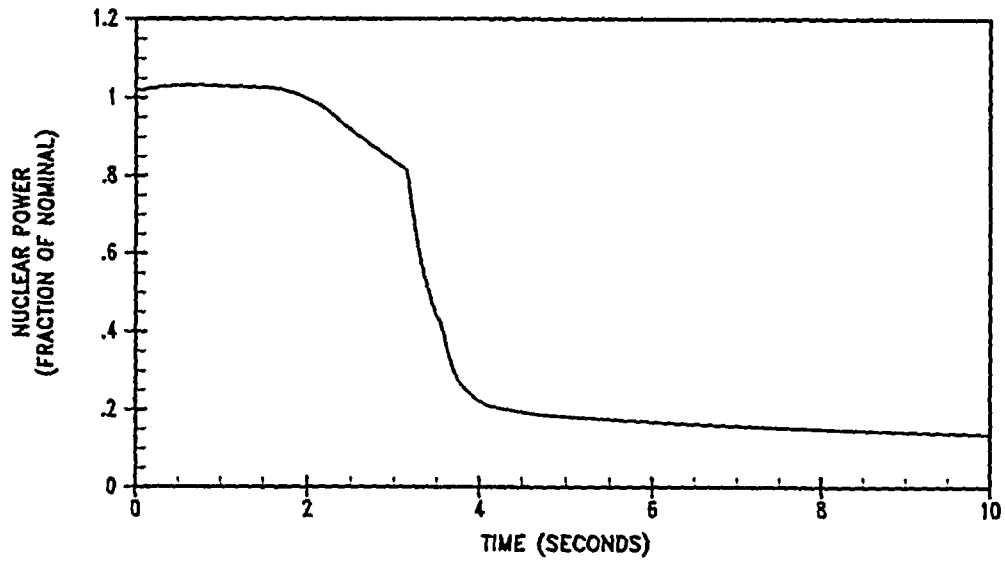


Figure 6.3-62
Locked Rotor/Shaft Break
(Nuclear Power and RCS Pressure vs. Time)

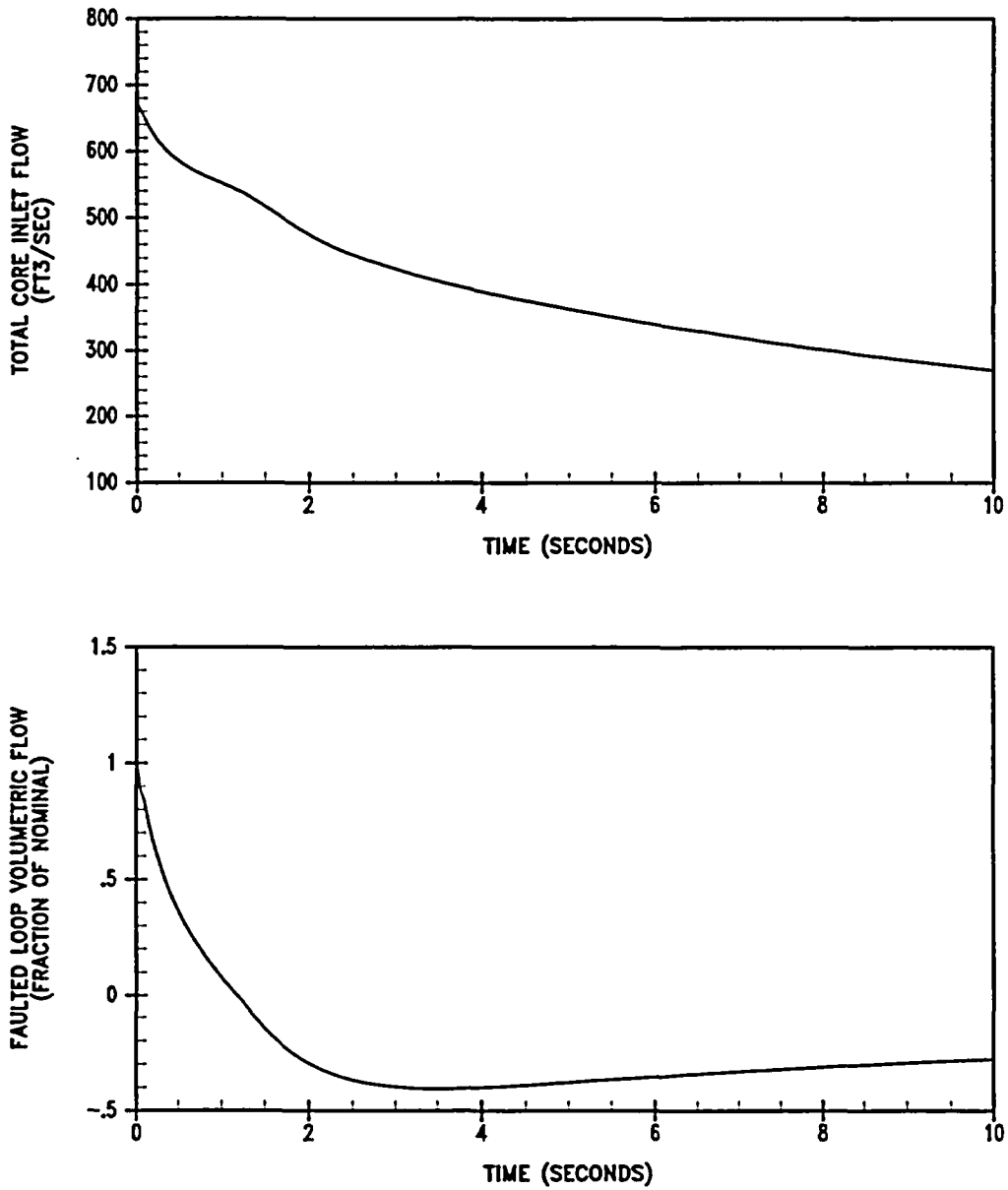


Figure 6.3-63
Locked Rotor/Shaft Break
(Total Core Flow and Faulted Loop Flow vs. Time)

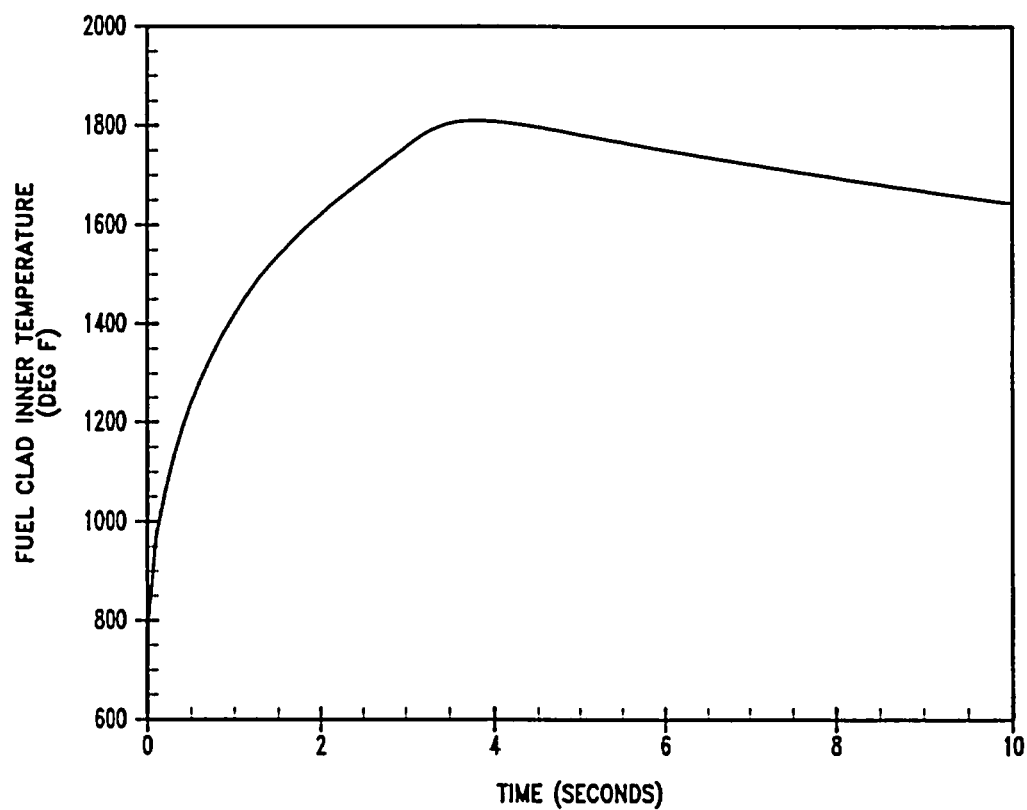


Figure 6.3-64
Locked Rotor/Shaft Break
(Fuel Clad Inner Temperature vs. Time)

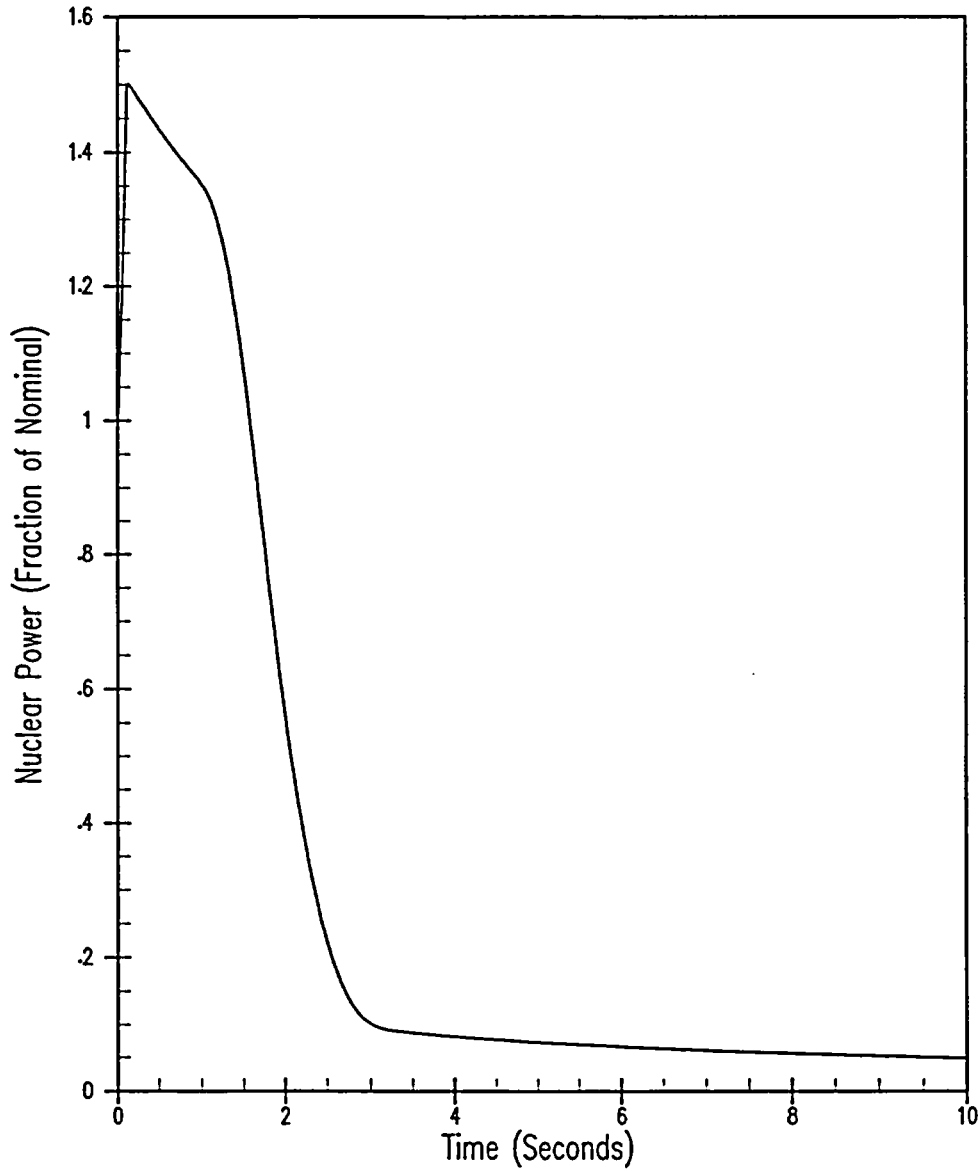


Figure 6.3-65
BOL HFP RCCA Ejection (Nuclear Power vs. Time)

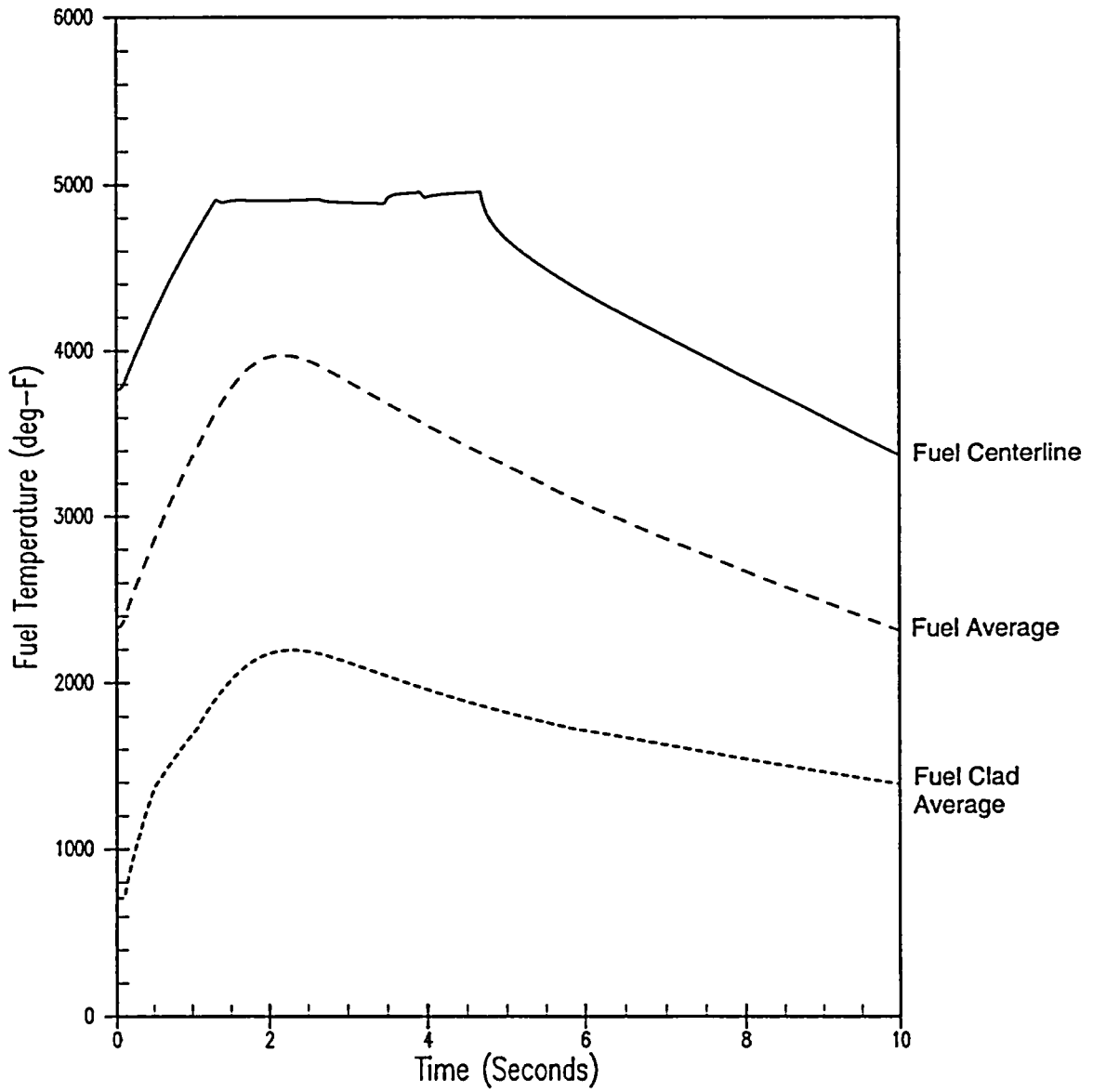


Figure 6.3-66
BOL HFP RCCA Ejection (Hot Spot Fuel and Clad Temperatures vs. Time)

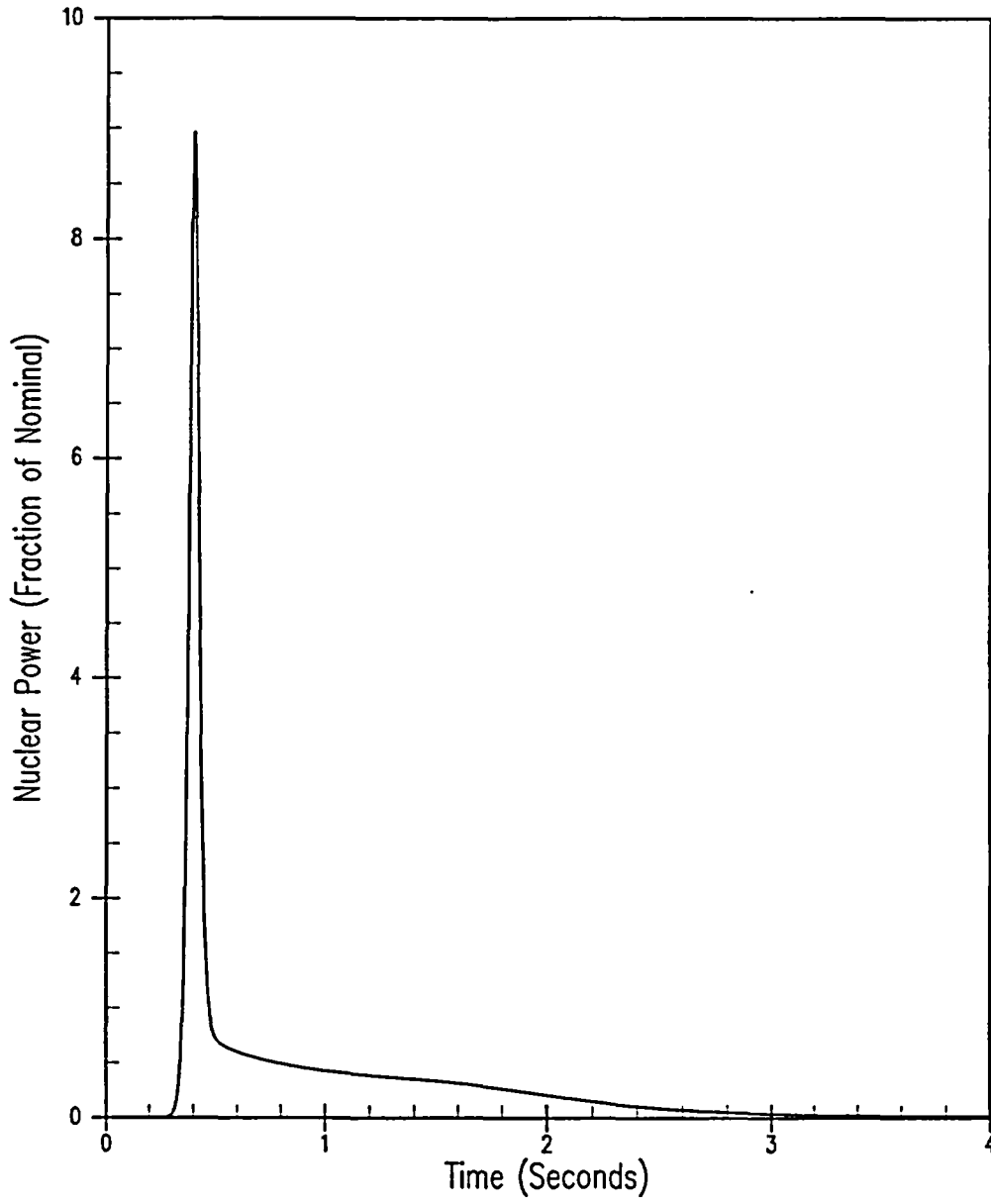


Figure 6.3-67
BOL HZP RCCA Ejection (Nuclear Power vs. Time)

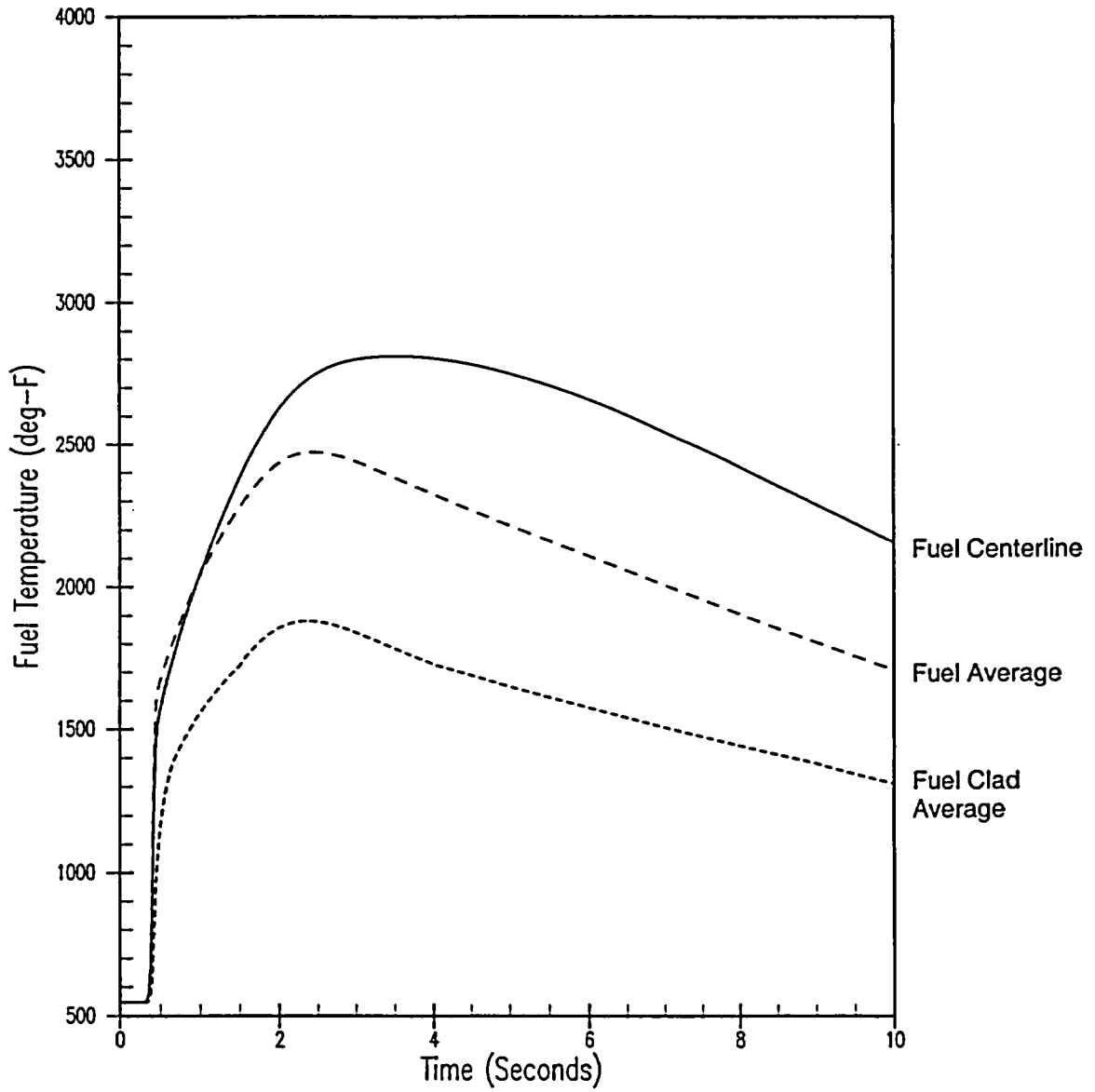


Figure 6.3-68
BOL HZP RCCA Ejection (Hot Spot Fuel and Clad Temperatures vs. Time)

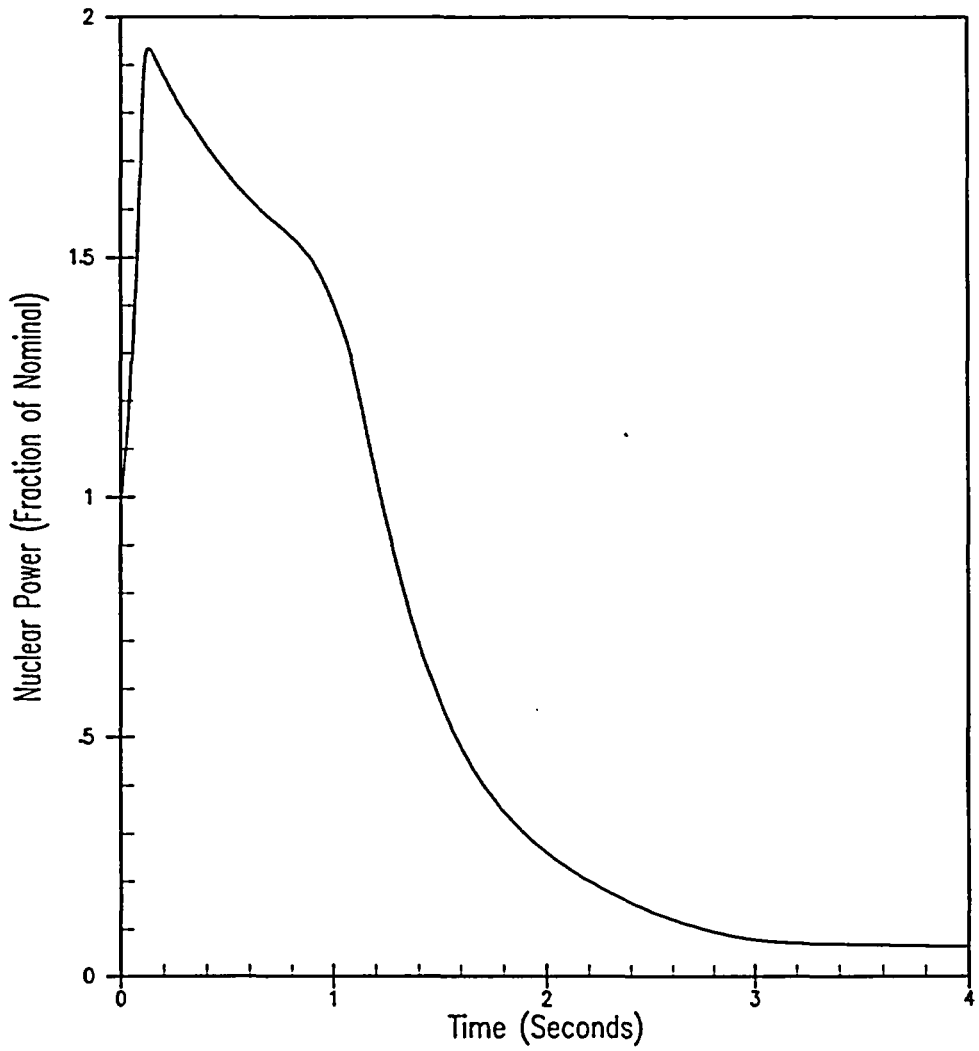


Figure 6.3-69
EOL HFP RCCA Ejection (Nuclear Power vs. Time)

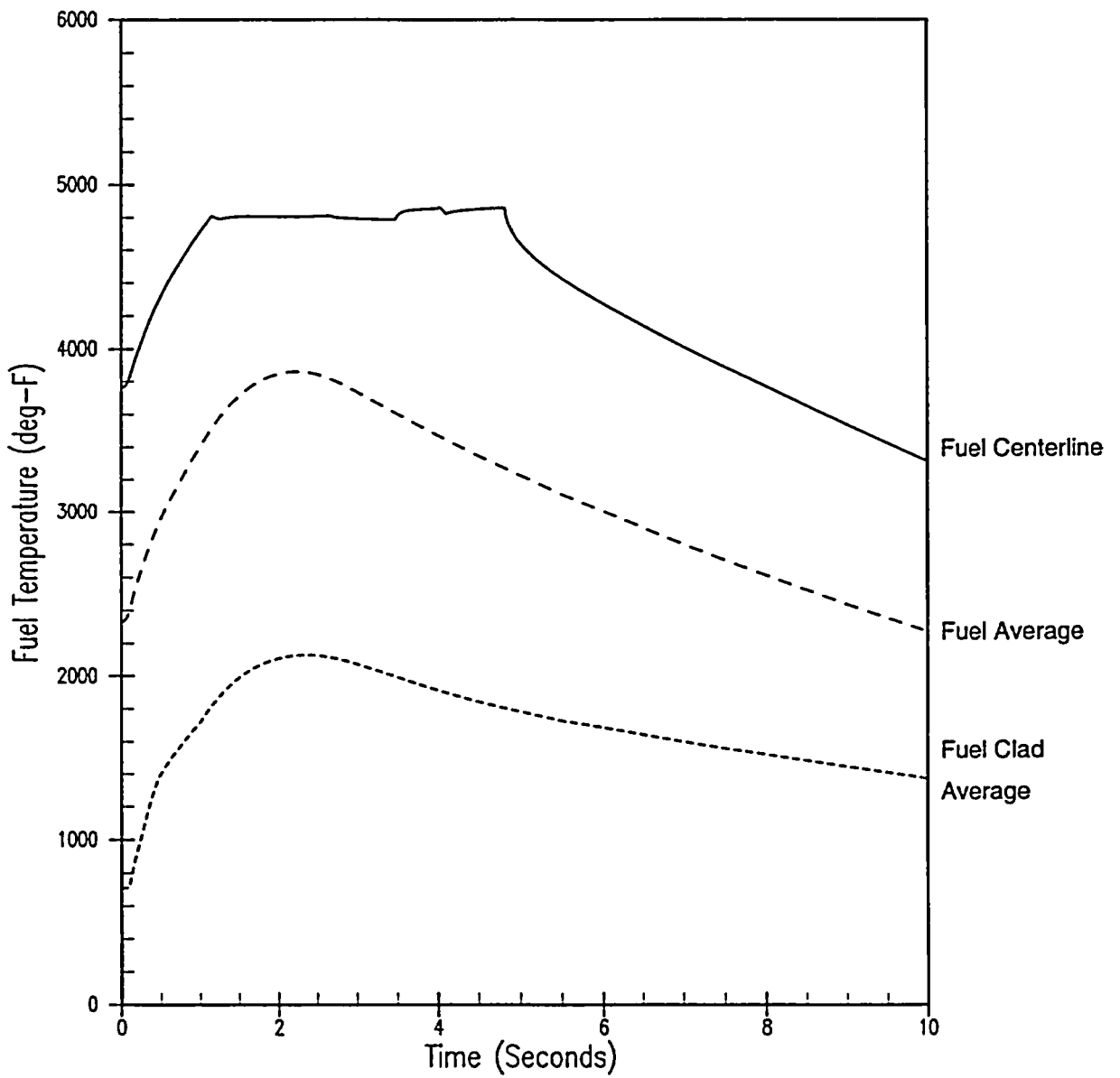


Figure 6.3-70
EOL HFP RCCA Ejection (Hot Spot Fuel and Clad Temperatures vs. Time)

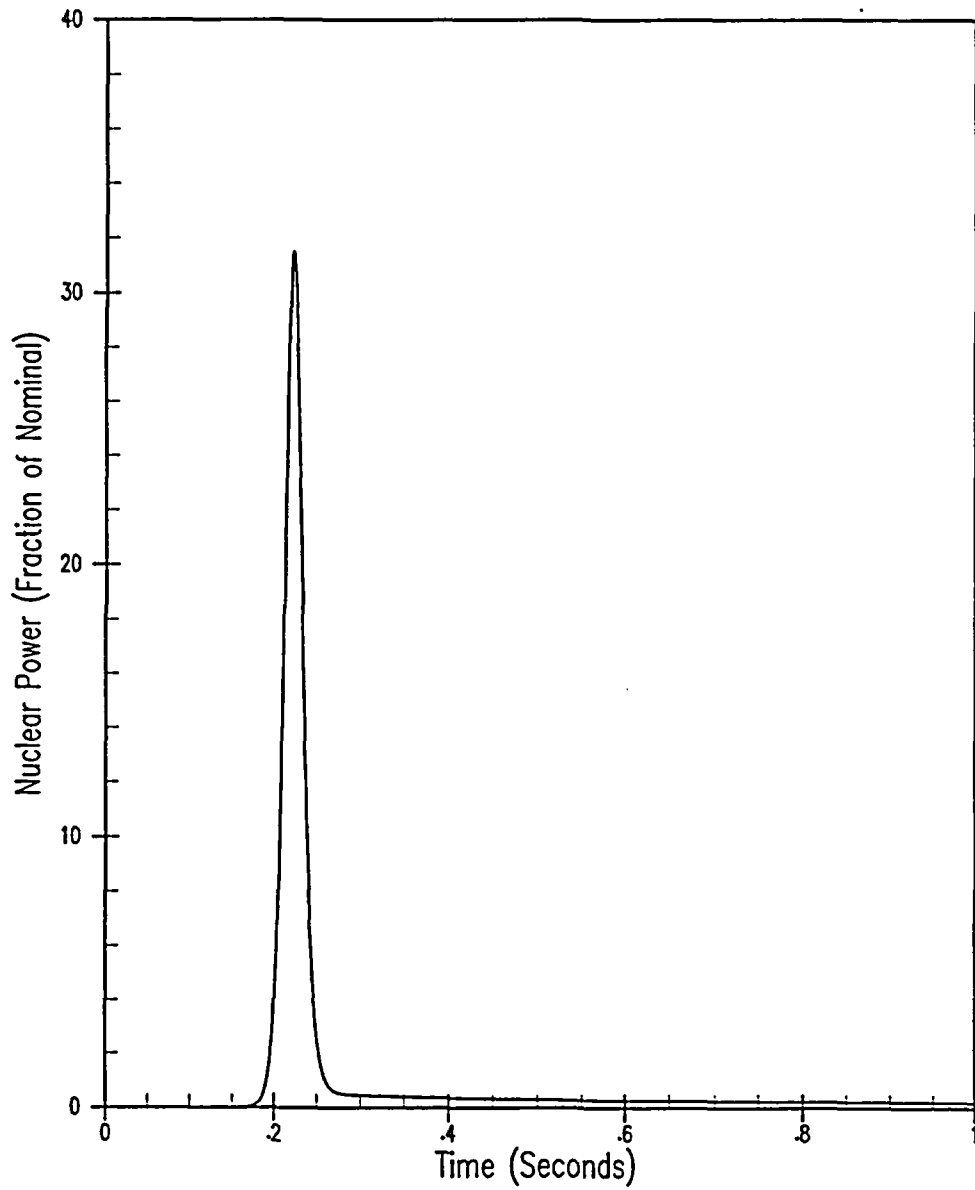


Figure 6.3-71
EOL HZP RCCA Ejection (Nuclear Power vs. Time)

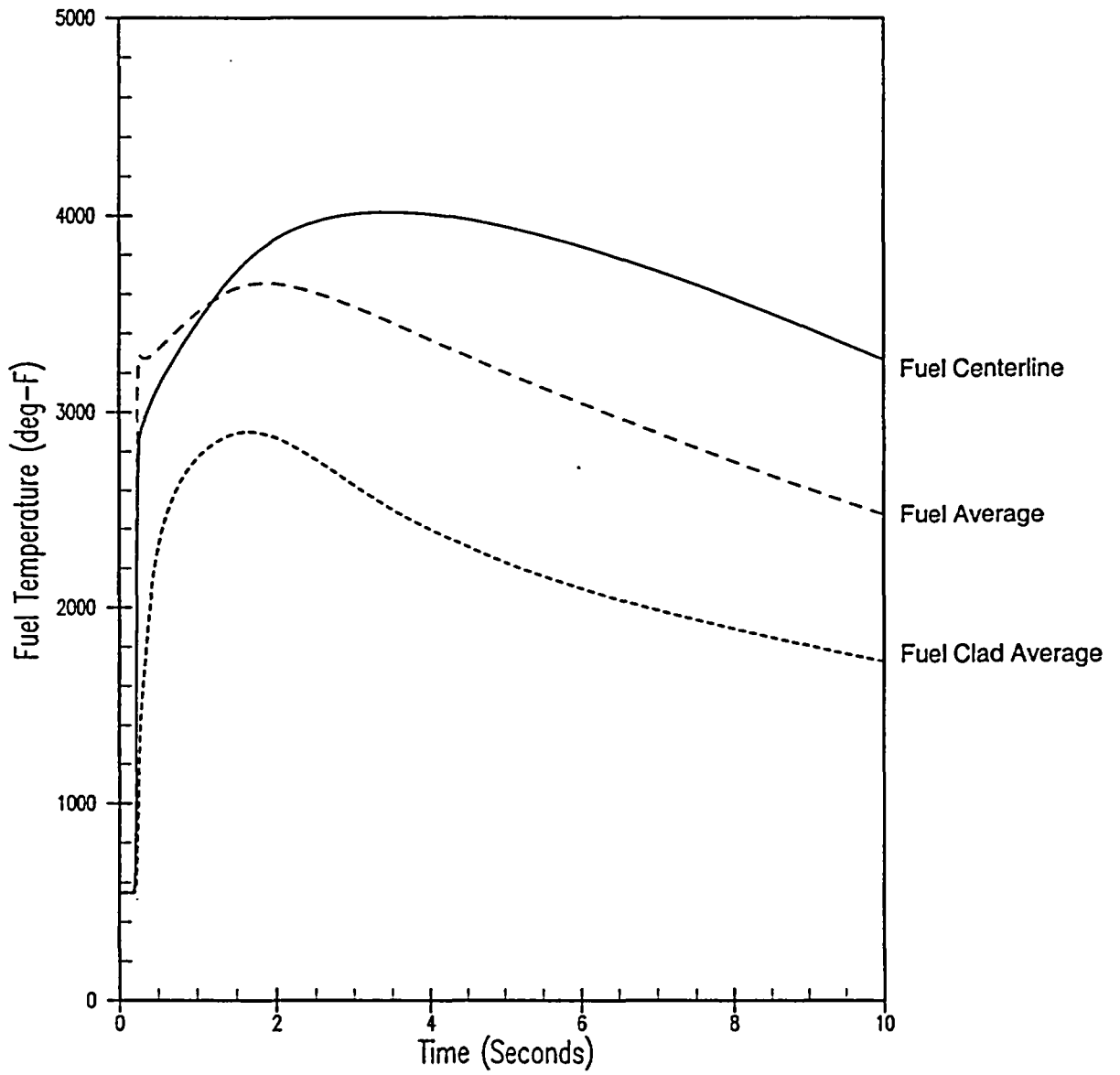


Figure 6.3-72
EOL HZP RCCA Ejection (Hot Spot Fuel and Clad Temperatures vs. Time)