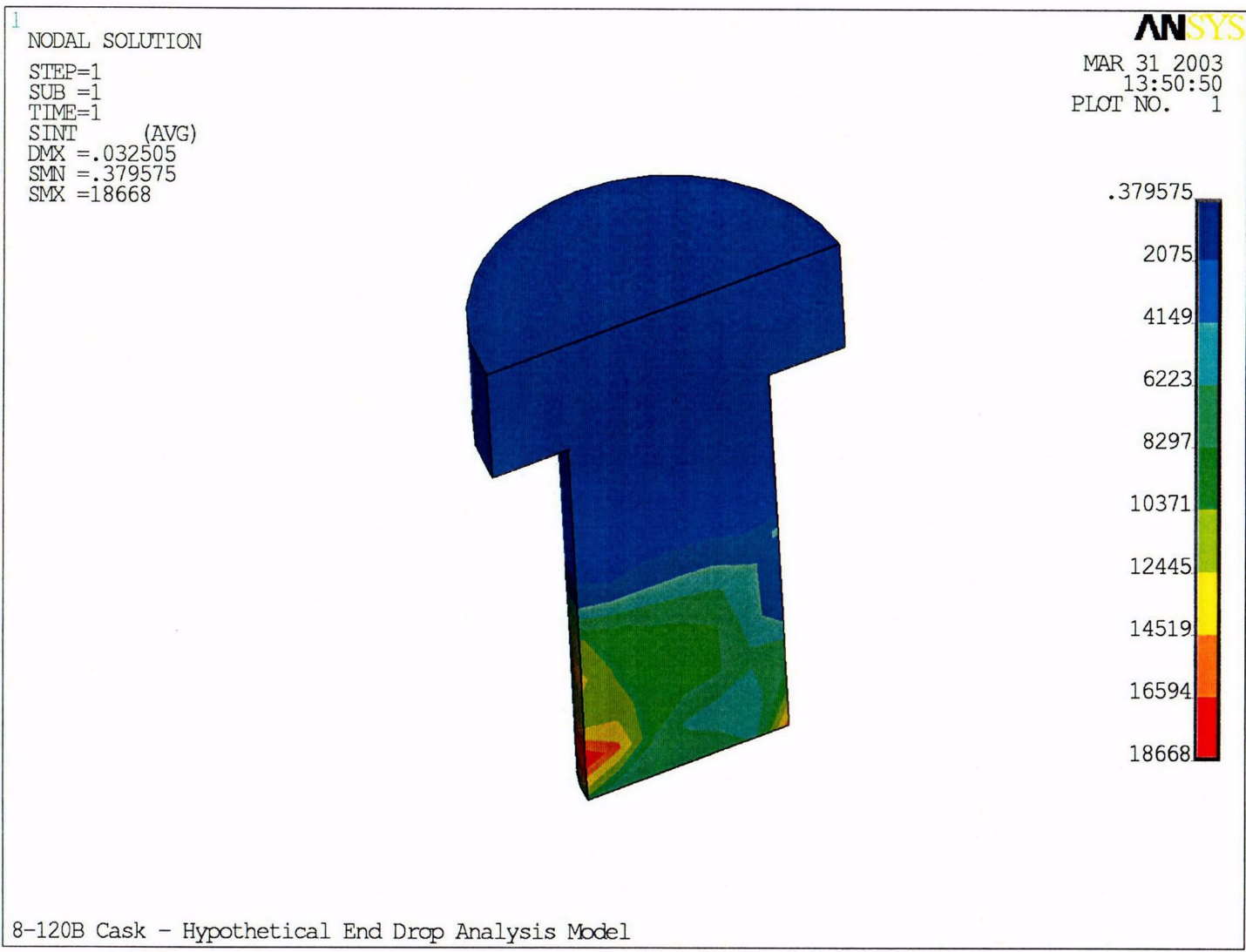


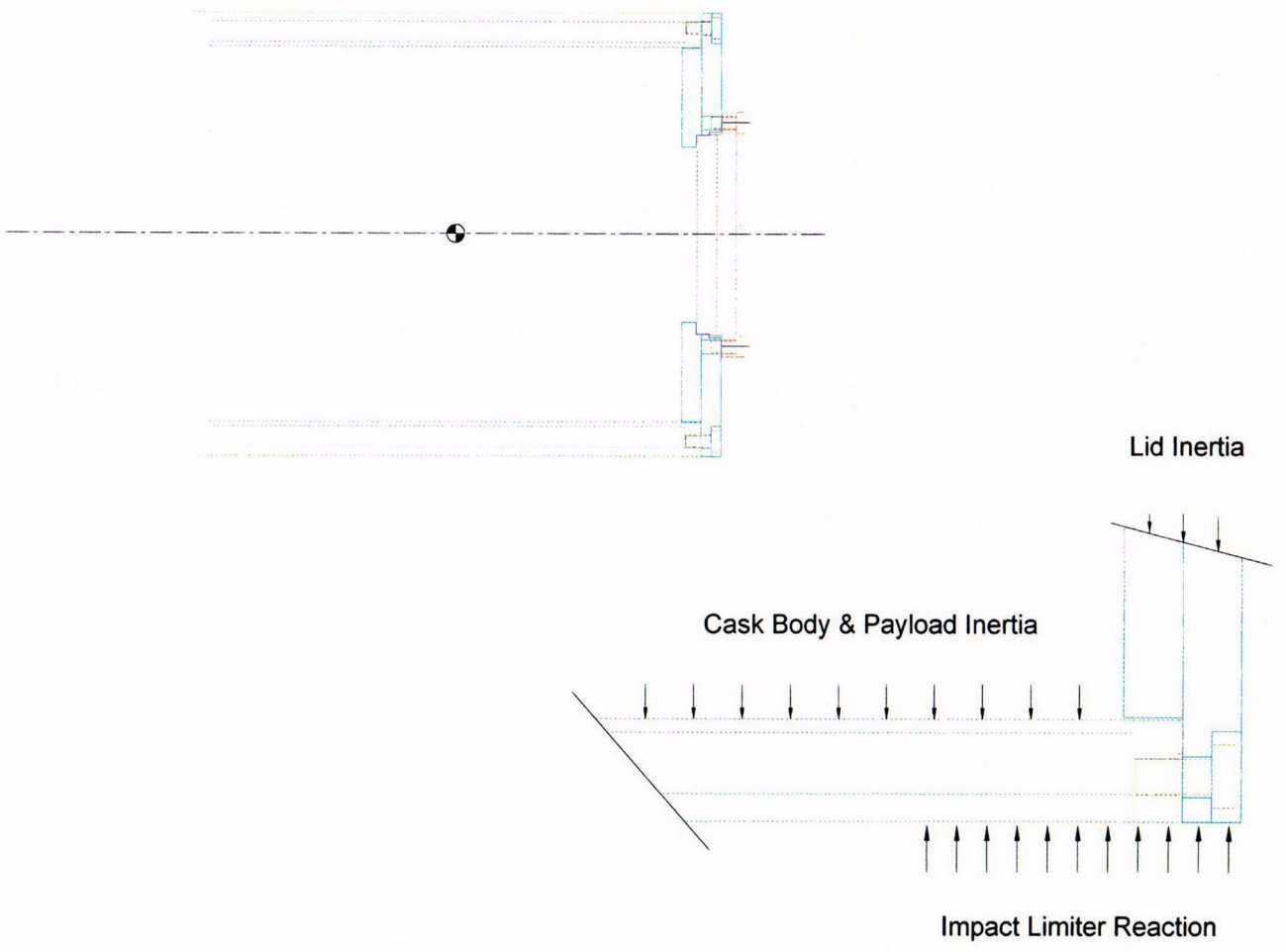
Figure 15  
 S.I. Distribution in the Inner & Outer Shells – Hypothetical Accident – End Drop



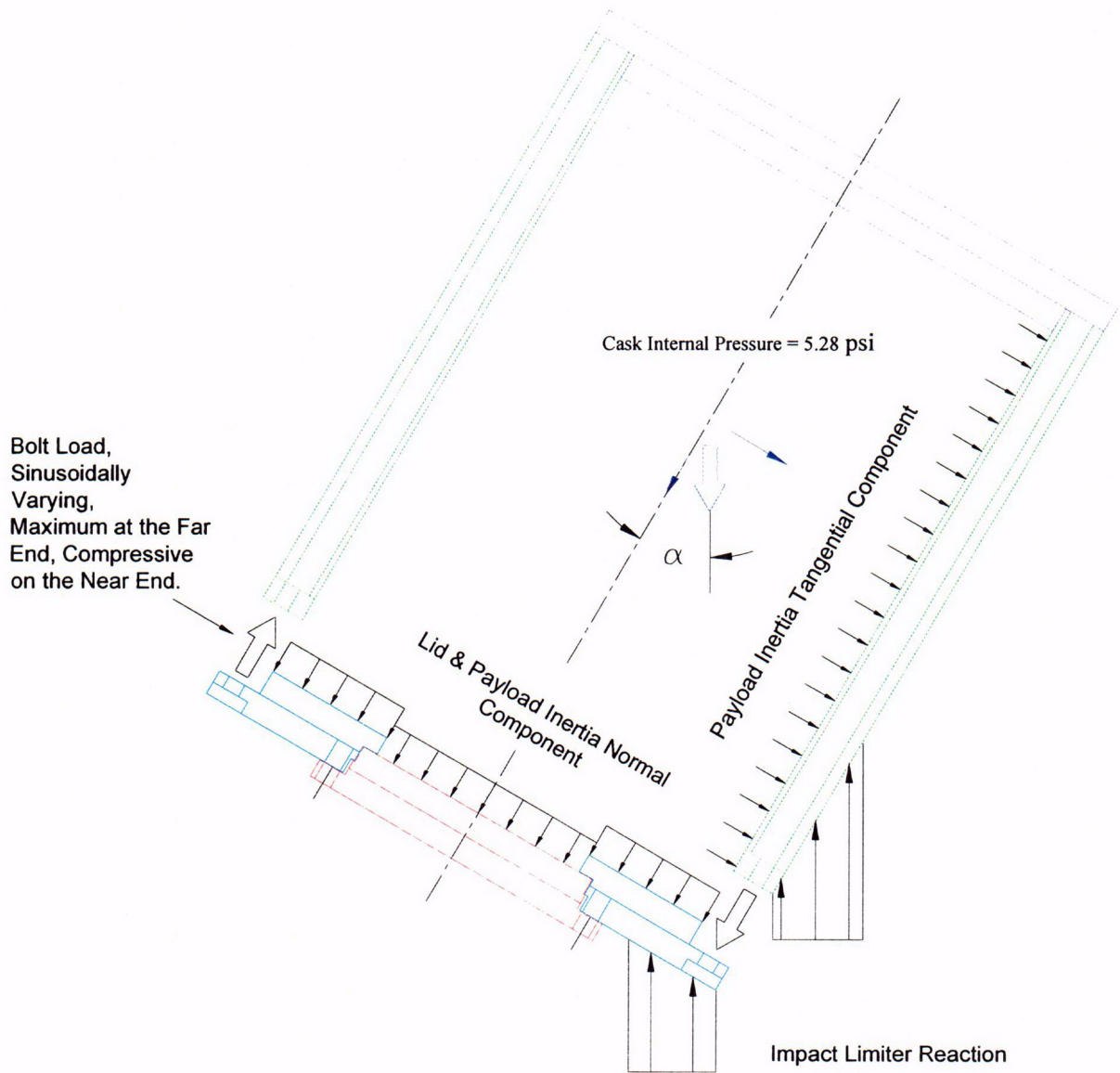
Title 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring  
Calc. No. ST-432 Rev. 0 Sheet 36 of 56

Figure 16  
S.I. Distribution in the Primary Lid Bolts – Hypothetical Accident – End Drop

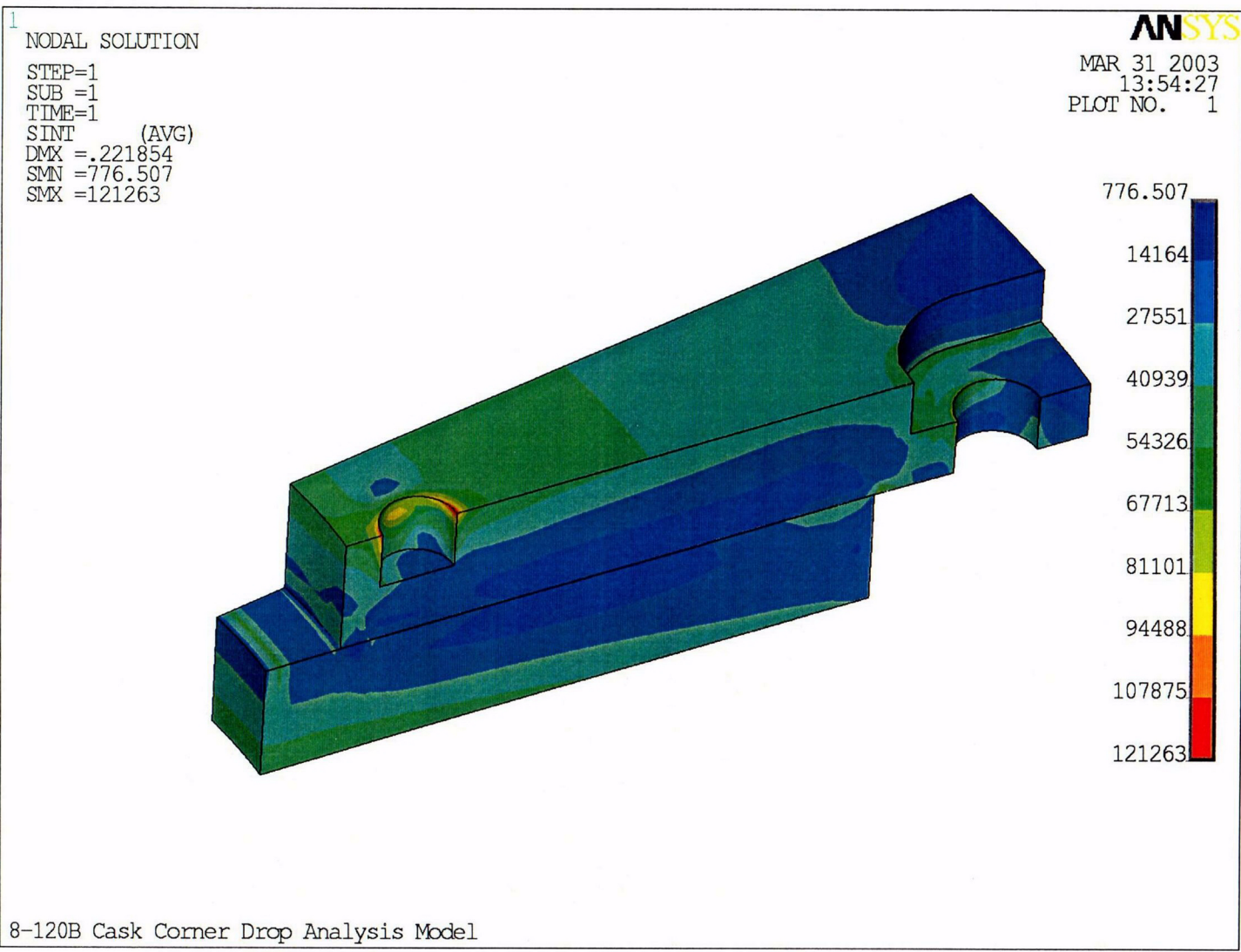




**Figure 17**  
Inertia Load and Impact Limiter Reaction on the Cask During Side Drop



**Figure 18**  
Inertia Load and Impact Limiter Reaction on the cask During Corner Drop



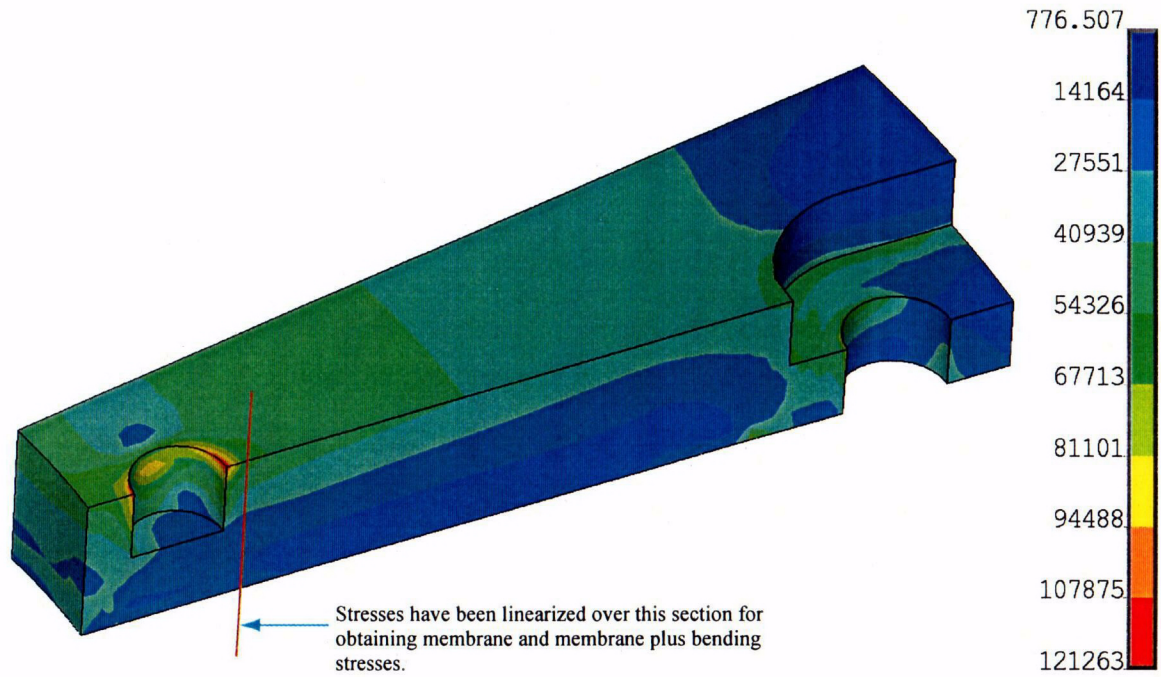
Title 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring  
Calc. No. ST-432 Rev. 0 Sheet 39 of 56

Figure 19  
S.I. Distribution in the Primary Lid – Hypothetical Accident – Corner Drop



1 NODAL SOLUTION  
 STEP=1  
 SUB =1  
 TIME=1  
 SINT (AVG)  
 DMX =.208084  
 SMN =776.507  
 SMX =121263

**ANSYS**  
 APR 1 2003  
 11:14:28  
 PLOT NO. 1



8-120B Cask Corner Drop Analysis Model

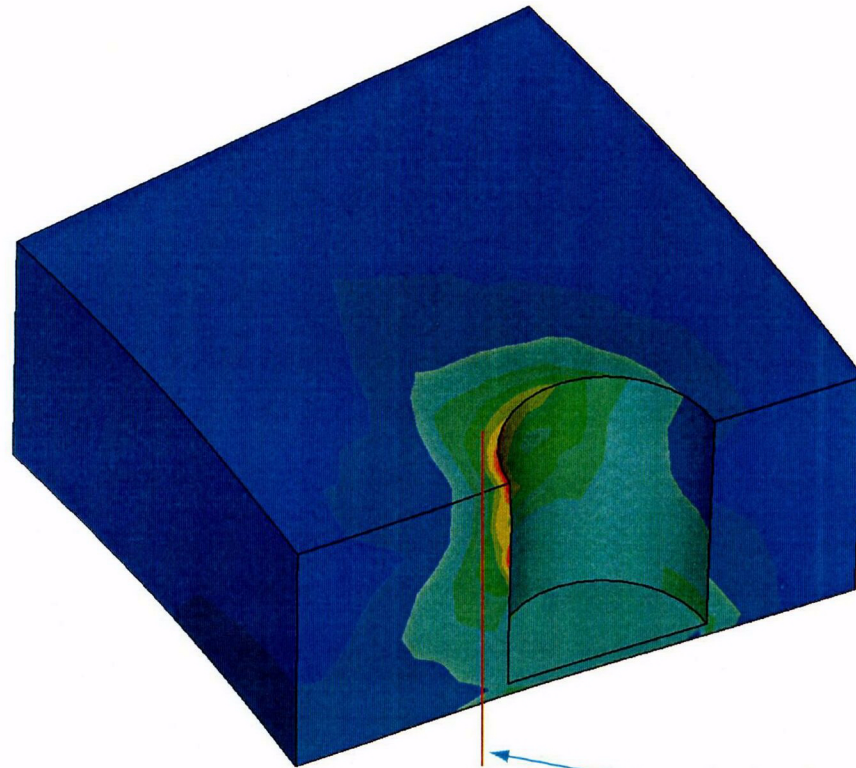
Title 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring  
 Calc. No. ST-432 Rev. 0 Sheet 40 of 56

**Figure 20**  
Location of Stress Linearization in the Primary Lid – Hypothetical Accident – Corner Drop

Revision 6  
 April 2003  
 C20

1 NODAL SOLUTION  
 STEP=1  
 SUB =1  
 TIME=1  
 SINT (AVG)  
 DMX =.015765  
 SMN =476.699  
 SMX =88998

**ANSYS**  
 MAR 31 2003  
 13:59:31  
 PLOT NO. 1



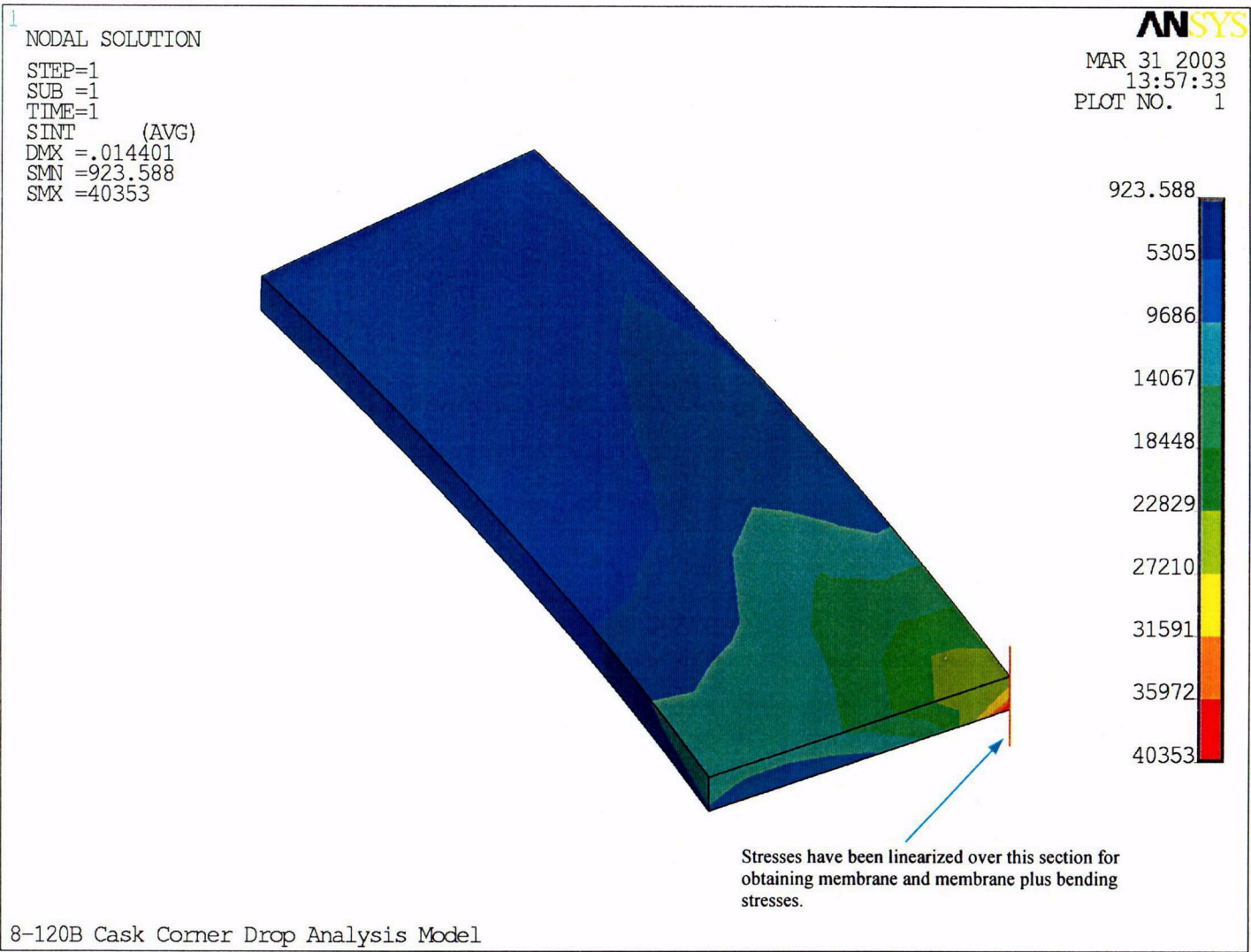
476.699  
 10312  
 20148  
 29984  
 39820  
 49655  
 59491  
 69327  
 79162  
 88998

Stresses have been linearized over this section for obtaining membrane and membrane plus bending stresses.

8-120B Cask Corner Drop Analysis Model

Title 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring  
 Calc. No. ST-432 Rev. 0 Sheet 41 of 56

Figure 21  
S.I. Distribution in the Bolting Ring – Hypothetical Accident – Corner Drop

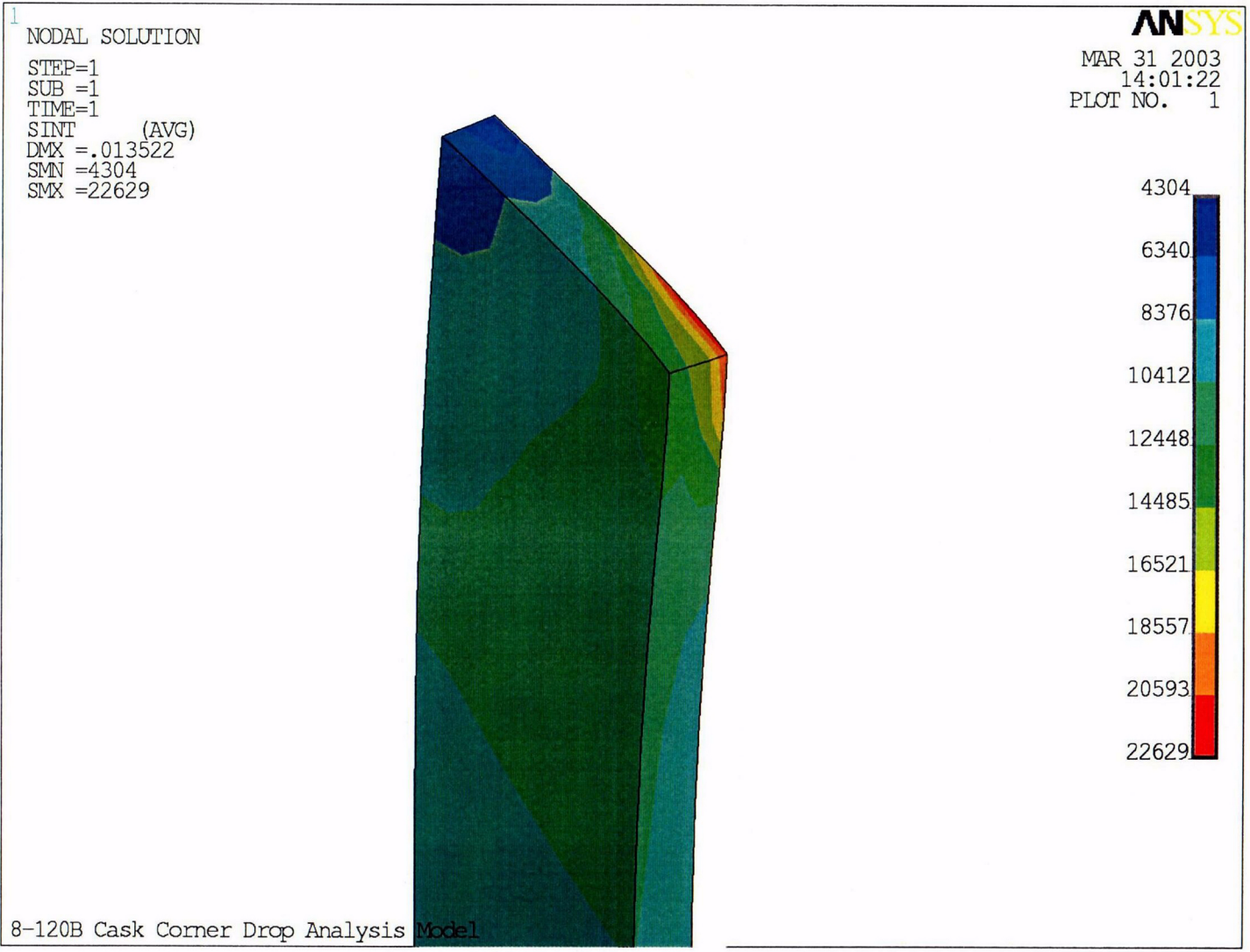


Title 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring

Calc. No. ST-432 Rev. 0 Sheet 42 of 56

Figure 22  
S.I. Distribution in the Seal Plate – Hypothetical Accident – Corner Drop





Title 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring

Calc. No. ST-432 Rev. 0 Sheet 43 of 56

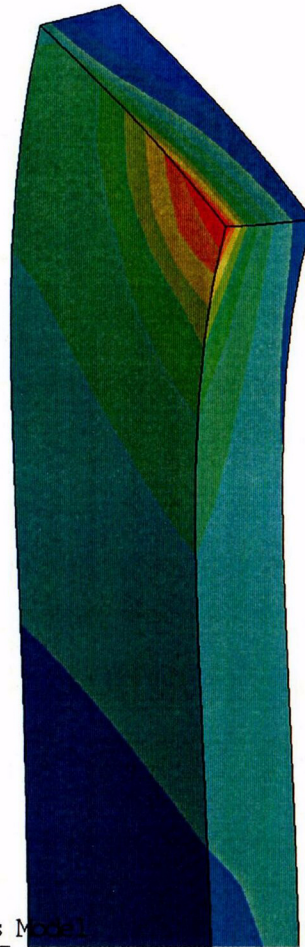
Figure 23  
S.I. Distribution in the Inner Shell – Hypothetical Accident – Corner Drop

Revision 6  
 April 2003

C23

1  
NODAL SOLUTION  
STEP=1  
SUB =1  
TIME=1  
SINT (AVG)  
DMX =.009914  
SMN =571.319  
SMX =27138

ANSYS  
MAR 31 2003  
14:02:52  
PLOT NO. 1

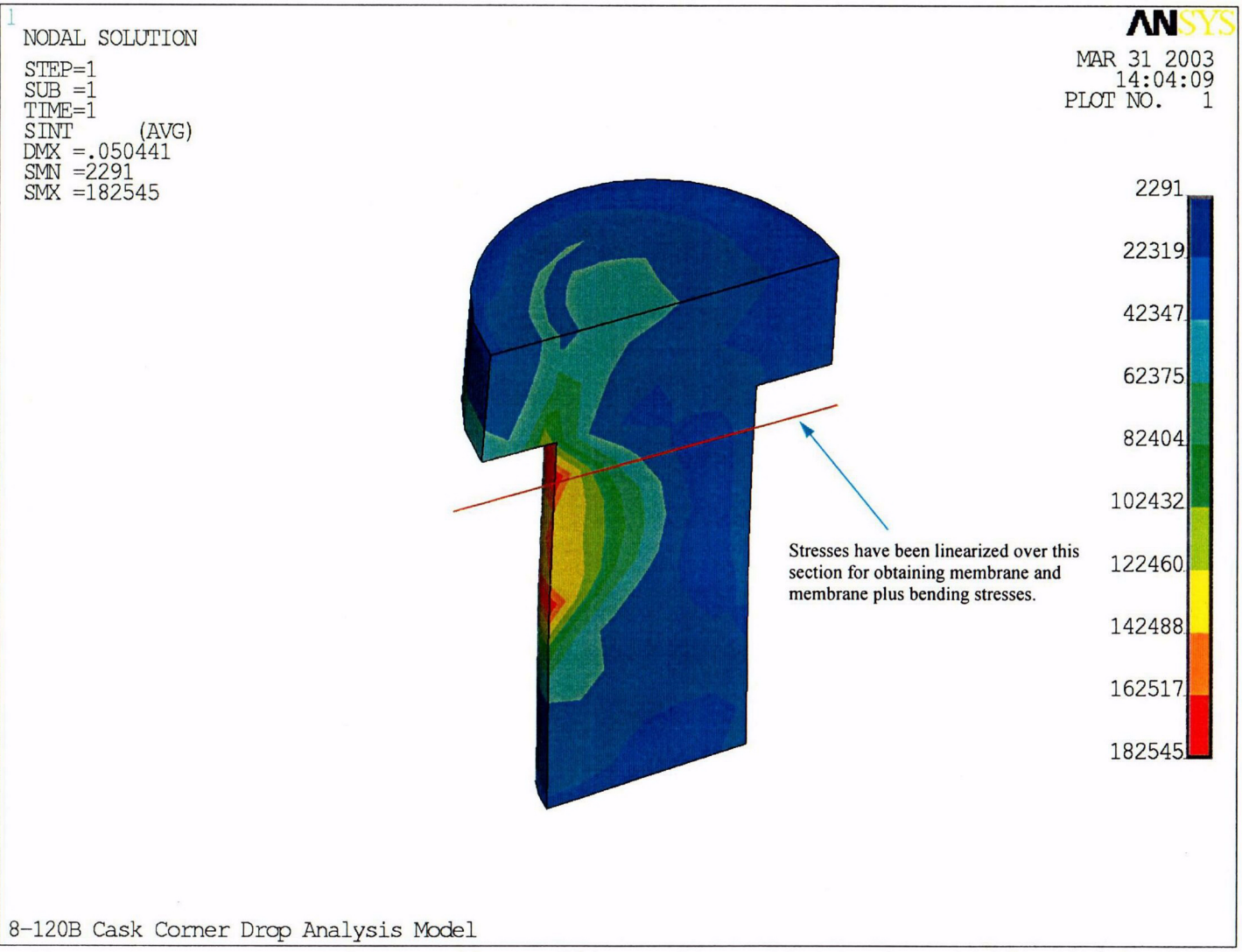


571.319  
3523  
6475  
9427  
12379  
15331  
18282  
21234  
24186  
27138

8-120B Cask Corner Drop Analysis Model

Title 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring  
Calc. No. ST-432 Rev. 0 Sheet 44 of 56

Figure 24  
S.I. Distribution in the Outer Shell – Hypothetical Accident – Corner Drop



Title 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring

Calc. No. ST-432 Rev. 0 Sheet 45 of 56

**Figure 25**  
S.I. Distribution in the Lid Bolts – Hypothetical Accident – Corner Drop

C25



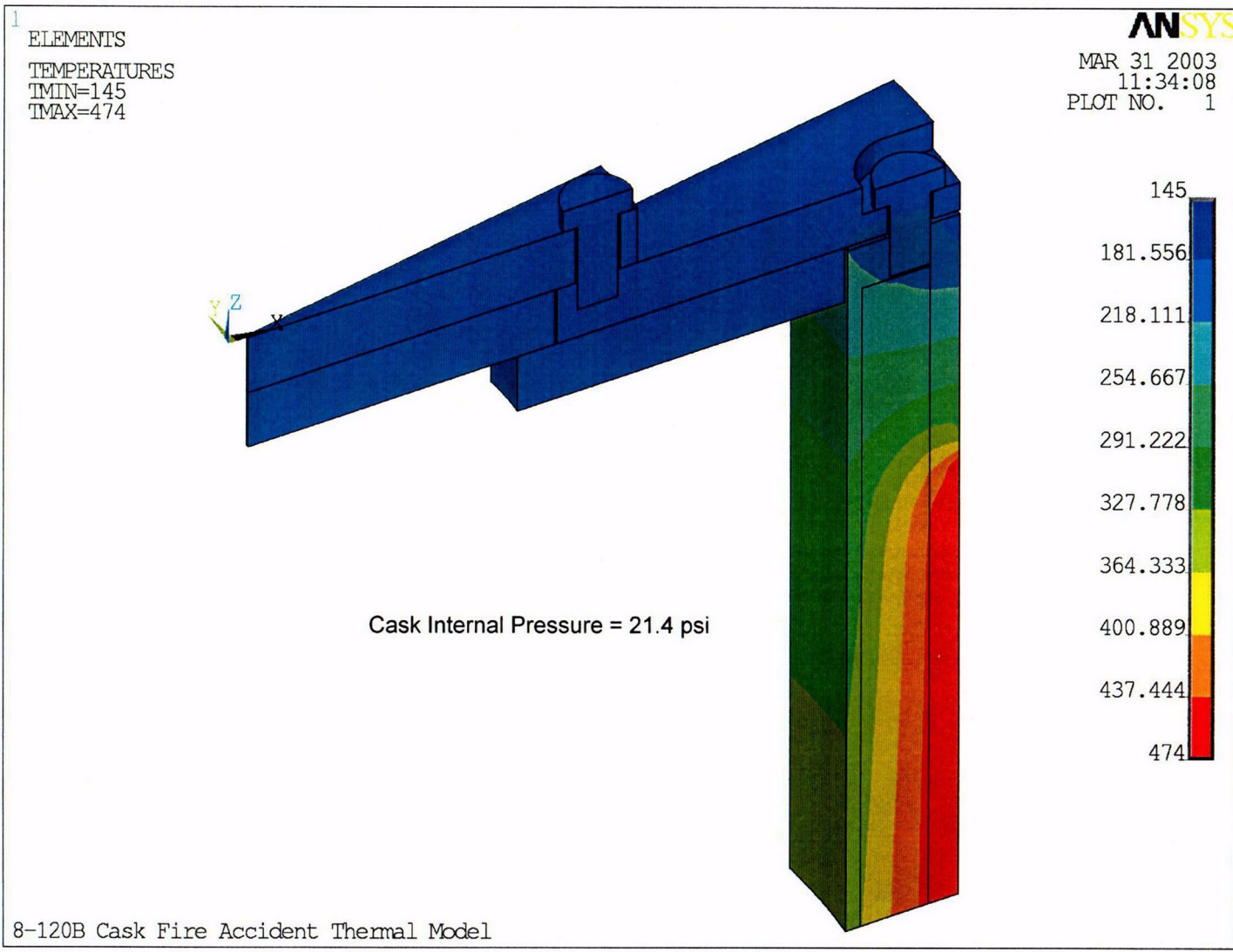
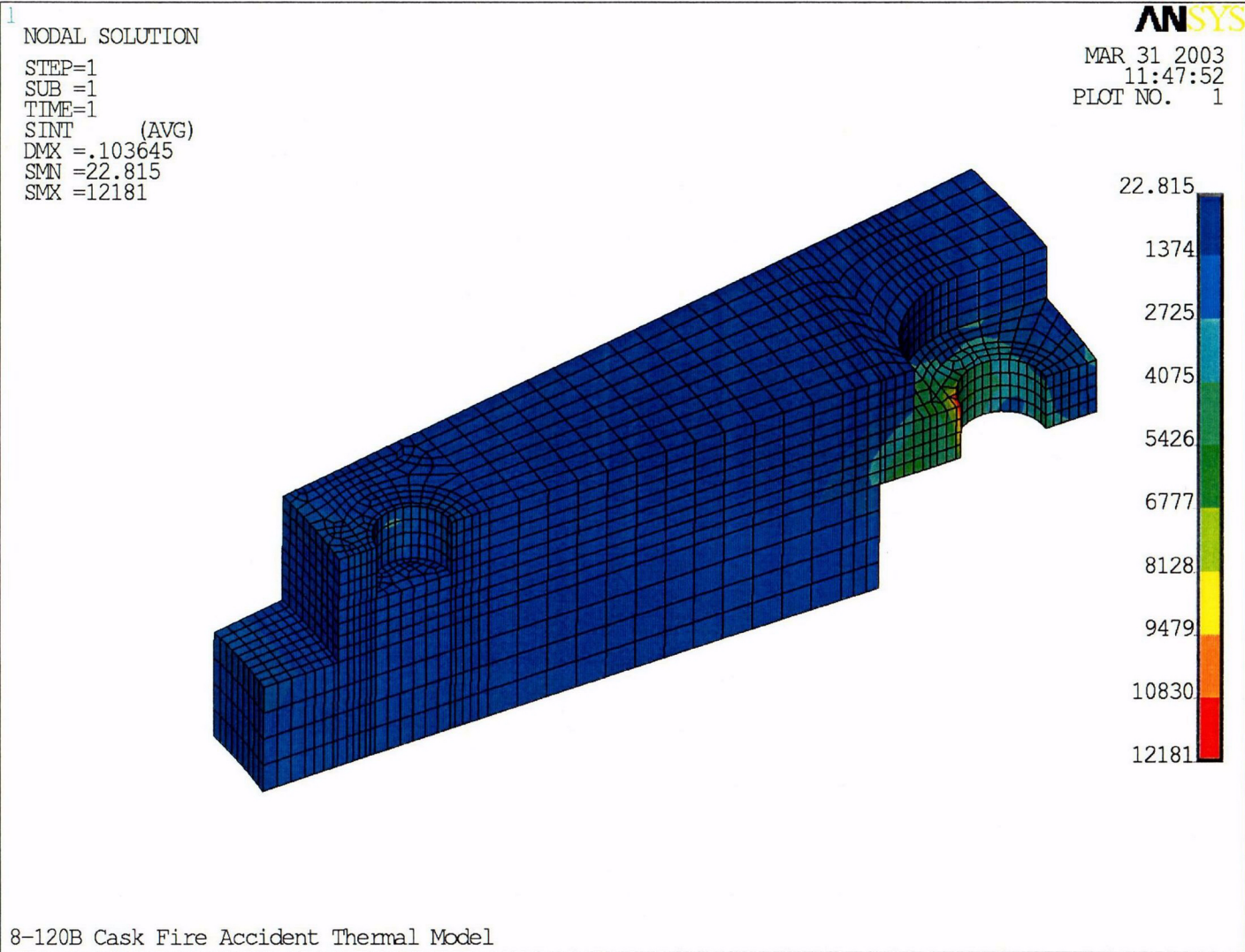


Figure 26  
Temperature Distribution in the Cask Under Hypothetical Fire Accident

C210

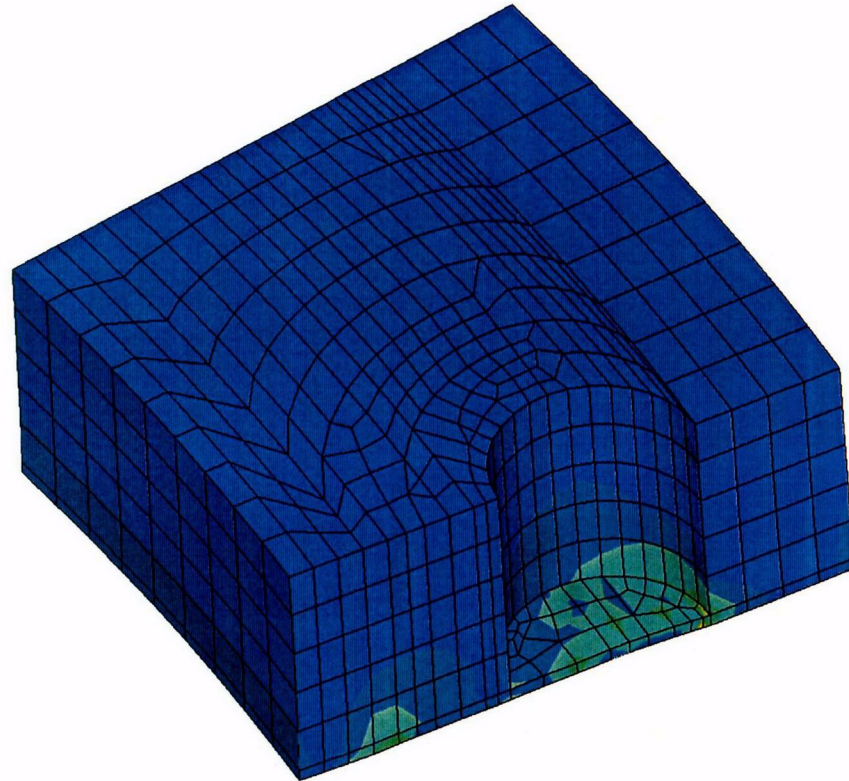


Title 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring  
Calc. No. ST-432 Rev. 0 Sheet 47 of 56

Figure 27  
S.I. Distribution in the Primary Lid – Hypothetical Fire Accident

1  
 NODAL SOLUTION  
 STEP=1  
 SUB =1  
 TIME=1  
 SINT (AVG)  
 DMX =.107516  
 SMN =231.398  
 SMX =106058

**ANSYS**  
 MAR 31 2003  
 13:26:48  
 PLOT NO. 1



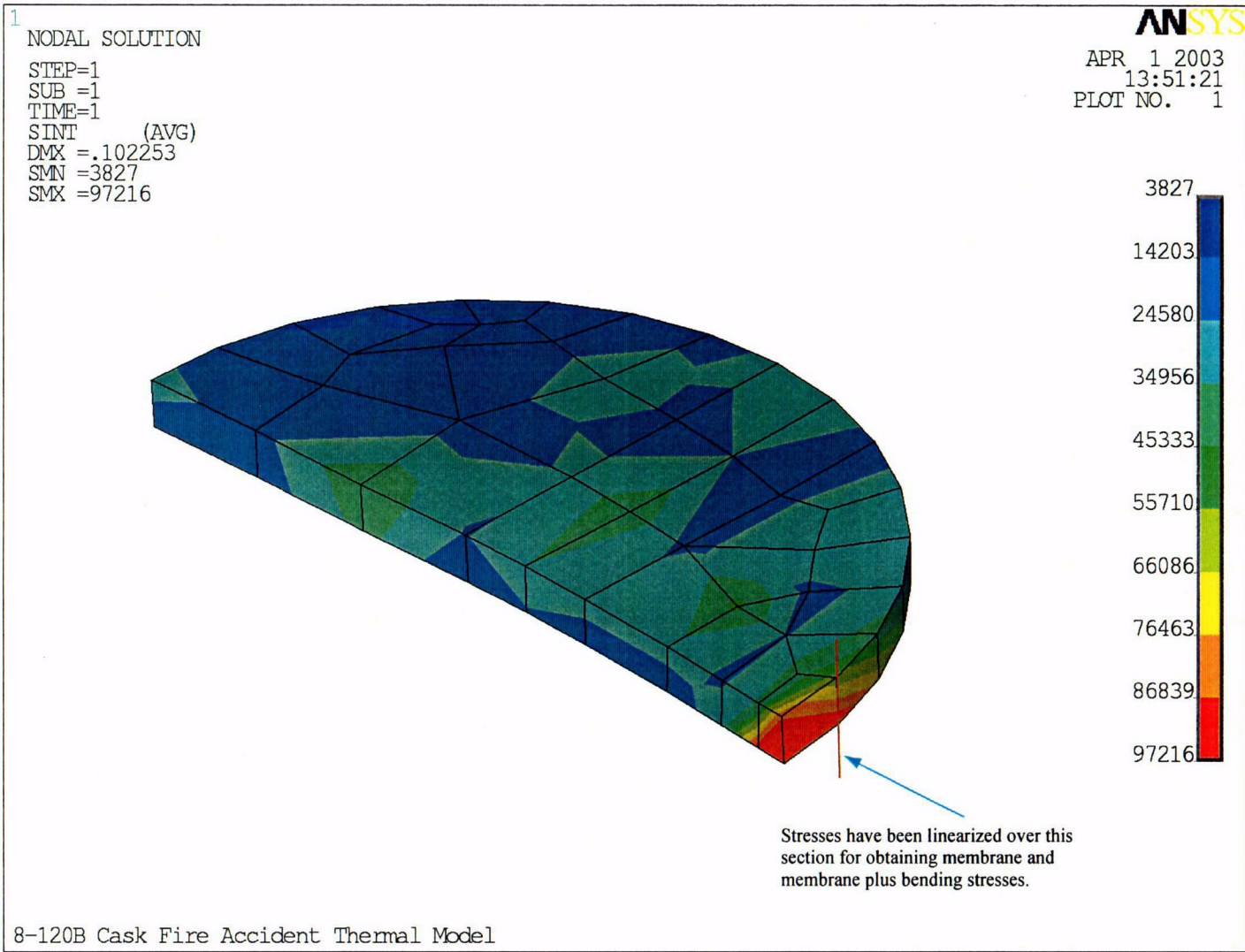
231.398  
 11990  
 23748  
 35507  
 47266  
 59024  
 70783  
 82541  
 94300  
 106058

8-120B Cask Fire Accident Thermal Model

**Title** 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring  
**Calc. No.** ST-432 **Rev.** 0 **Sheet** 48 of 56

**Figure 28**  
S.I. Distribution in the Bolting Ring – Hypothetical Fire Accident



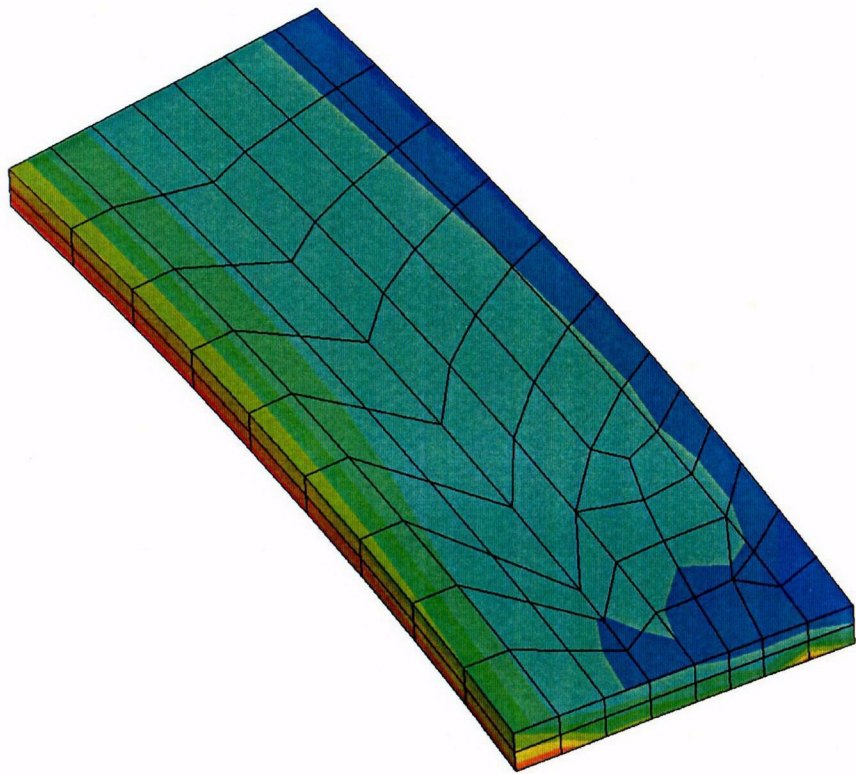


**Figure 29**  
Location of Stress Linearization in the Bolting Ring – Hypothetical Fire Accident

C29  
Revision 6  
April 2003

1  
 NODAL SOLUTION  
 STEP=1  
 SUB =1  
 TIME=1  
 SINT (AVG)  
 DMX =.096008  
 SMN =4598  
 SMX =14908

**ANSYS**  
 MAR 31 2003  
 11:55:26  
 PLOT NO. 1



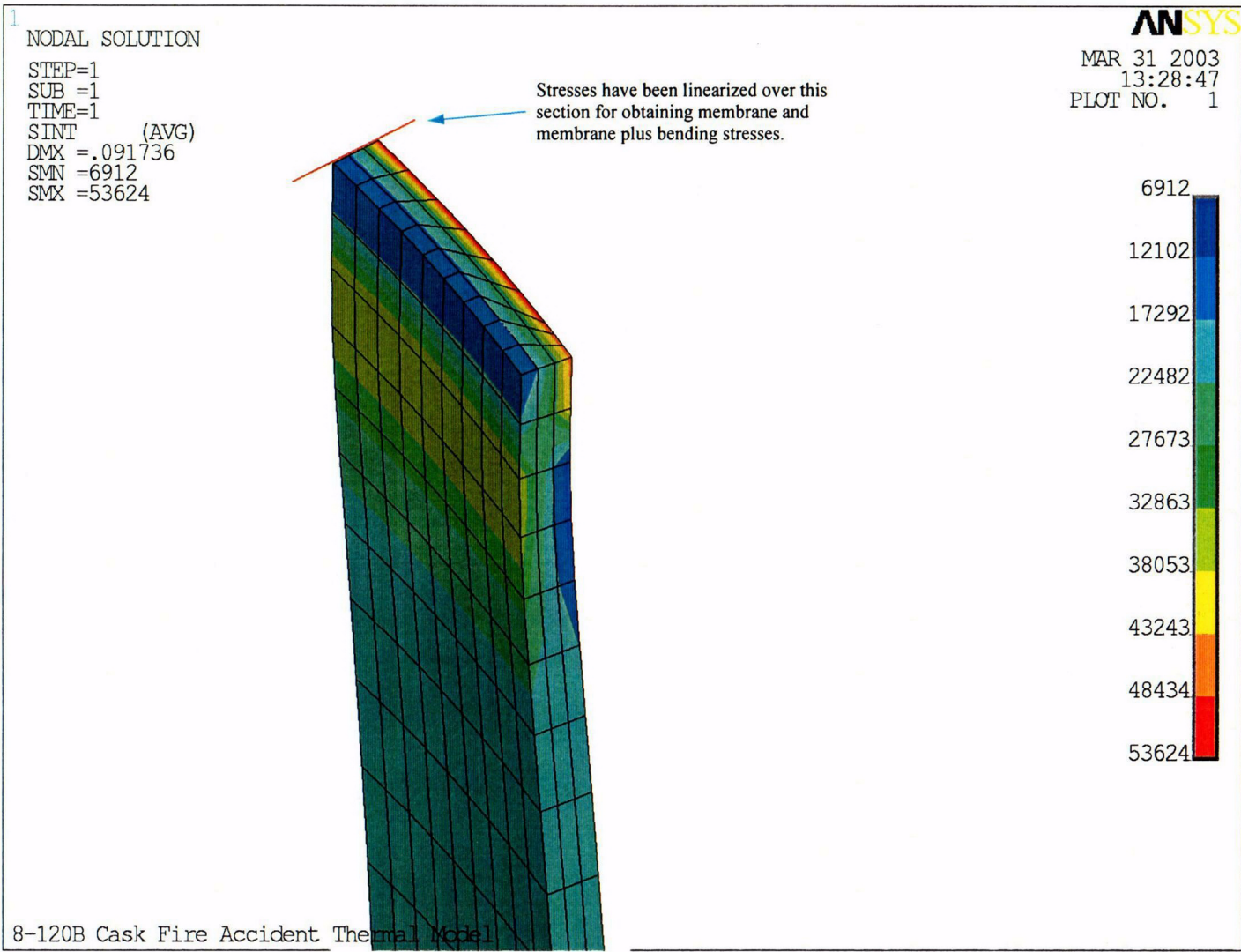
4598  
 5743  
 6889  
 8035  
 9180  
 10326  
 11471  
 12617  
 13763  
 14908

8-120B Cask Fire Accident Thermal Model

Title 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring  
 Calc. No. ST-432 Rev. 0 Sheet 50 of 56

**Figure 30**  
**S.I. Distribution in the Seal Plate – Hypothetical Fire Accident**

C30  
 Revision 6  
 April 2003



**Figure 31**  
S.I. Distribution in the Inner Shell – Hypothetical Fire Accident



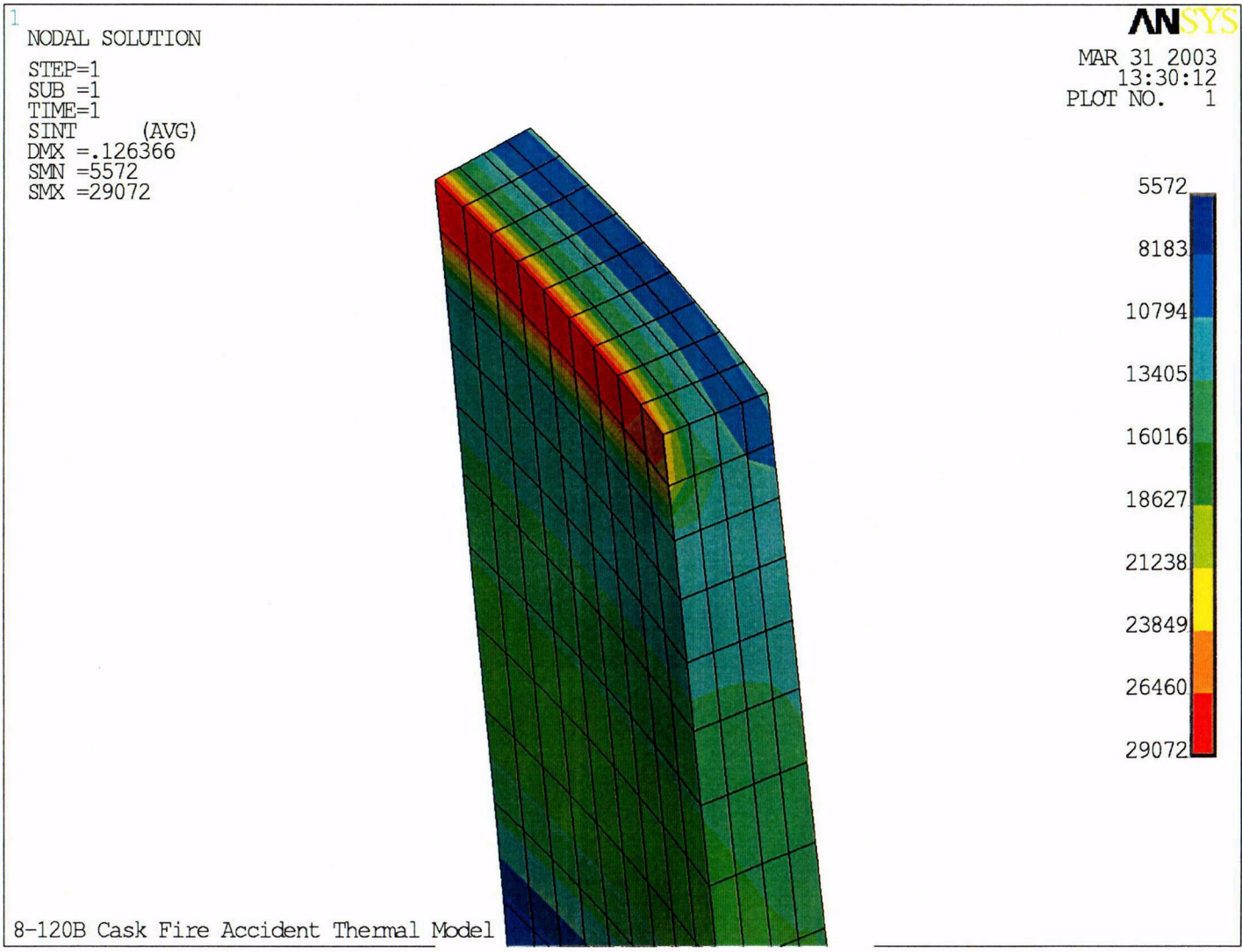
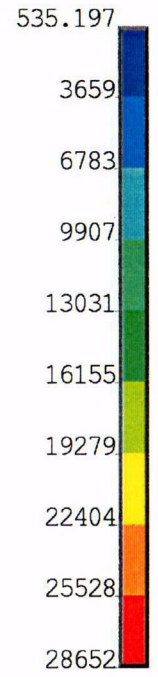
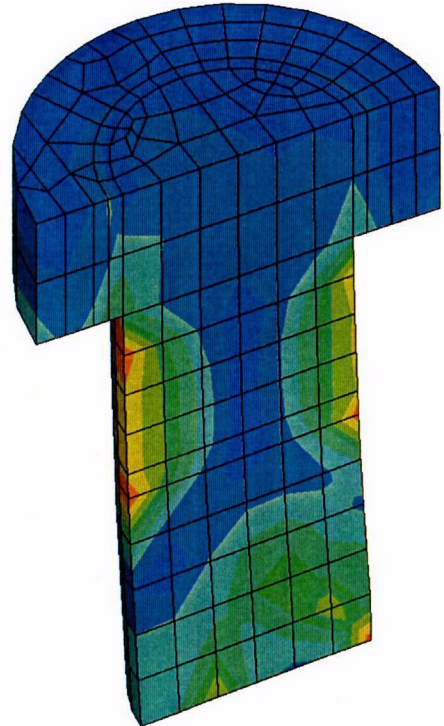


Figure 32  
S.I. Distribution in the Outer Shell – Hypothetical Fire Accident

1 NODAL SOLUTION  
STEP=1  
SUB =1  
TIME=1  
SINT (AVG)  
DMX =.101749  
SMN =535.197  
SMX =28652

ANSYS  
MAR 31 2003  
13:32:00  
PLOT NO. 1



8-120B Cask Fire Accident Thermal Model

Title 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring  
Calc. No. ST-432 Rev. 0 Sheet 53 of 56

Figure 33  
S.I. Distribution in the Lid Bolts – Hypothetical Fire Accident

C33  
Revision 6  
April 2003

Title 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring

Calc. No. ST-432 Rev. 0

Sheet 54 of 56

Appendix A  
CNS 8-120B Cask Analyses  
ANSYS Finite Element Model Details  
(8 Pages)



CNS 8-120B Cask Analyses  
ANSYS Finite Element Model Details

**Database Listing**

G L O B A L   S T A T U S

ANSYS - Engineering Analysis System                      Apr 03, 2003                      11:03  
Release 6.1                      00222442                      INTEL NT                      Version

Current working directory: E:\Ansys Analyses\8-120B\Thermal\Normal\Stress

MENULIST File: E:\Program Files\Ansys Inc\ANSYS61\docu\english\menulist61.ans

Product(s) enabled: ANSYS/Mechanical/Emag 3D

Total connect time. . . . . 2 hours 47 minutes  
Total CP usage. . . . . 0 hours 0 minutes 12.3 seconds

J O B   I N F O R M A T I O N -----

8-120B Cask Normal Conditions - Thermal Stress Analysis

Current jobname . . . . . .file  
Initial jobname . . . . . .file

Units . . . . . .unknown

	Available	Used
Scratch Memory Space. . . . .	32.000 mb	1.283 mb ( 4.0%)
Database space . . . . .	8191.750 mb	74.019 mb ( 0.9%)

Note! Used database space exceeds memory resident size of 32.0 mb.  
Page file is used.

User menu file in use . . .E:\Program Files\Ansys  
Inc\ANSYS61\DOCU\english\uidl\UIMENU.GRN  
User menu file in use . . .E:\Program Files\Ansys  
Inc\ANSYS61\DOCU\english\uidl\UIFUNC1.GRN  
User menu file in use . . .E:\Program Files\Ansys  
Inc\ANSYS61\DOCU\english\uidl\UIFUNC2.GRN  
User menu file in use . . .E:\Program Files\Ansys  
Inc\ANSYS61\DOCU\english\uidl\MECHTOOL.AUI  
Beta features . . . . .are not shown in the user interface

M O D E L   I N F O R M A T I O N -----

Solid model summary:

	Largest Number	Number Defined	Number Selected
Keypoints . . . . .	708	708	708

Lines . . . . .	1319	1319	1319
Areas . . . . .	802	802	802
Volumes . . . . .	163	163	163

Finite element model summary:

	Largest Number	Number Defined	Number Selected
Nodes . . . . .	20635	20635	20635
Elements . . . . .	16247	16247	16247
Element types . . . . .	3	3	n.a.
Real constant sets . . . . .	15	5	n.a.
Material property sets . . . . .	6	6	n.a.
Coupling . . . . .	7072	1561	n.a.
Constraint equations . . . . .	0	0	n.a.
Master DOFs . . . . .	0	0	n.a.
Dynamic gap conditions . . . . .	0	0	n.a.

BOUNDARY CONDITION INFORMATION -----

	Number Defined	X	Y	Z
Constraints on nodes . . . . .	5966			
Constraints on keypoints . . . . .	0			
Constraints on lines . . . . .	0			
Constraints on areas . . . . .	0			
Forces on nodes . . . . .	0			
Forces on keypoints . . . . .	0			
Surface loads on elements . . . . .	767			
Number of element flagged surfaces . . . . .	0			
Surface loads on lines . . . . .	0			
Surface loads on areas . . . . .	0			
Body loads on elements . . . . .	0			
Body loads on nodes . . . . .	20635			
Body loads on keypoints . . . . .	0			
Temperatures				
Uniform temperature . . . . .	70.000			
Reference temperature . . . . .	70.000			
Offset from absolute scale . . . . .	0.000			
Linear acceleration . . . . .	0.0000	0.0000	0.0000	0.0000
Angular velocity (about global CS) . . . . .	0.0000	0.0000	0.0000	0.0000
Angular acceleration (about global CS) . . . . .	0.0000	0.0000	0.0000	0.0000
Location of reference CS . . . . .	0.0000	0.0000	0.0000	0.0000
Angular velocity (about reference CS) . . . . .	0.0000	0.0000	0.0000	0.0000
Angular acceleration (about reference CS) . . . . .	0.0000	0.0000	0.0000	0.0000

ROUTINE INFORMATION -----

Current routine . . . . .Preprocessing (PREP7)

```

Active coordinate system . . . . . 12 (Cylindrical)
Display coordinate system. . . . . 0 (Cartesian)

Current element attributes:
  Type number . . . . . 3 (CONTAC49)
  Real number . . . . . 13
  Material number . . . . . 5
  Element coordinate system number. . 0

Current mesher type. . . . . .free mesher

Current element meshing shape 2D . . .use default element shape.
Current element meshing shape 3D . . .use hexahedra.

SmrtSize Level . . . . . OFF

Global element size. . . . . 0 divisions per line

Active coordinate system . . . . . 12 (Cylindrical)
Display coordinate system. . . . . 0 (Cartesian)

Analysis type. . . . . .Static (steady-state)

Active options for this analysis type:
  Large deformation effects . . . . .Not included
  Plasticity. . . . .Not included
  Creep . . . . .Not included
  Equation solver to use. . . . .Program Chosen

Results file . . . . . .file.rst

Load step number . . . . . 2

Number of substeps:
  Starting number of substeps . . . . 1
  Maximum number of substeps. . . . . 100
  Minimum number of substeps. . . . . 1
  Step change boundary conditions . .No
  
```

---



---

**Element Type Listing**

---



---

```

LIST ELEMENT TYPES FROM      1 TO      3 BY      1

ELEMENT TYPE      1 IS SOLID45      3-D STRUCTURAL SOLID      INOPR
KEYOPT(1-12)=    0 0 0 0 0 0 0 0 0 0 0 0      0

ELEMENT TYPE      2 IS COMBIN40      COMBINATION      INOPR
  
```



```

KEYOPT(1-12)=    0  0  3   0  0  0   0  0  0   0  0  0   0
ELEMENT TYPE     3 IS CONTAC49      3-D POINT-TO-SURFACE CONTACT  INOPR
KEYOPT(1-12)=    0  0  0   0  0  0   0  0  0   0  0  0   0

CURRENT NODAL DOF SET IS  UX    UY    UZ
THREE-DIMENSIONAL MODEL

```

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---

### Real Constant Listing

---



---

```

LIST REAL SETS      1 TO      15 BY      1
REAL CONSTANT SET   7 ITEMS  1 TO   6
  1.0000      0.0000      0.0000      0.10000E-03      0.0000      0.10000E+09
REAL CONSTANT SET  12 ITEMS  1 TO   6
  0.0000      0.0000      0.0000      0.10000E-03      0.0000      0.10000E+07
REAL CONSTANT SET  13 ITEMS  1 TO   6
  0.10000E+07  0.0000      0.10000E-02  0.0000      5.0000      0.0000
REAL CONSTANT SET  14 ITEMS  1 TO   6
  0.0000      0.0000      0.0000      0.25000      0.0000      0.10000E+07
REAL CONSTANT SET  15 ITEMS  1 TO   6
  0.0000      0.0000      0.0000      100.00      0.0000      0.0000
REAL CONSTANT SET  15 ITEMS  7 TO  12
  0.10000E-01  0.0000      0.0000      0.0000      0.0000      0.0000

```

---



---

### Material Properties Listing

---



---

```

LIST MATERIALS      1 TO      6 BY      1
PROPERTY= ALL
PROPERTY TABLE EX  MAT=      1 NUM. POINTS=  6
  TEMPERATURE      DATA  TEMPERATURE      DATA  TEMPERATURE      DATA
  70.000      0.27900E+08  100.00      0.27800E+08  200.00      0.27700E+08
  300.00      0.27400E+08  400.00      0.27000E+08  500.00      0.26400E+08

PROPERTY TABLE NUXY MAT=      1 NUM. POINTS=  6
  TEMPERATURE      DATA  TEMPERATURE      DATA  TEMPERATURE      DATA
  70.000      0.30000      100.00      0.30000      200.00      0.30000
  300.00      0.30000      400.00      0.30000      500.00      0.30000

PROPERTY TABLE ALPX MAT=      1 NUM. POINTS=  6
  TEMPERATURE      DATA  TEMPERATURE      DATA  TEMPERATURE      DATA

```

70.000	0.65000E-05	100.00	0.65000E-05	200.00	0.66700E-05
300.00	0.68700E-05	400.00	0.70700E-05	500.00	0.72500E-05

PROPERTY TABLE DENS MAT= 1 NUM. POINTS= 1

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
0.0000	0.28300				

PROPERTY TABLE KXX MAT= 1 NUM. POINTS= 16

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.81300E-03	100.00	0.80300E-03	200.00	0.77800E-03
300.00	0.74800E-03	400.00	0.71500E-03	500.00	0.67700E-03
600.00	0.64800E-03	700.00	0.61600E-03	800.00	0.58300E-03
900.00	0.55100E-03	1000.0	0.51900E-03	1100.0	0.48400E-03
1200.0	0.45100E-03	1300.0	0.41700E-03	1400.0	0.38000E-03
1500.0	0.36300E-03				

PROPERTY TABLE C MAT= 1 NUM. POINTS= 16

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.10330	100.00	0.10530	200.00	0.11210
300.00	0.11770	400.00	0.12340	500.00	0.12780
600.00	0.13220	700.00	0.13810	800.00	0.14520
900.00	0.15350	1000.0	0.16240	1100.0	0.17100
1200.0	0.18290	1300.0	0.20450	1400.0	0.40100
1500.0	0.19820				

PROPERTY TABLE EX MAT= 2 NUM. POINTS= 6

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.27900E+08	100.00	0.27800E+08	200.00	0.27700E+08
300.00	0.27400E+08	400.00	0.27000E+08	500.00	0.26400E+08

PROPERTY TABLE NUXY MAT= 2 NUM. POINTS= 6

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.30000	100.00	0.30000	200.00	0.30000
300.00	0.30000	400.00	0.30000	500.00	0.30000

PROPERTY TABLE ALPX MAT= 2 NUM. POINTS= 6

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.65000E-05	100.00	0.65000E-05	200.00	0.66700E-05
300.00	0.68700E-05	400.00	0.70700E-05	500.00	0.72500E-05

PROPERTY TABLE DENS MAT= 2 NUM. POINTS= 1

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
0.0000	0.28300				

PROPERTY TABLE KXX MAT= 2 NUM. POINTS= 16

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.81300E-03	100.00	0.80300E-03	200.00	0.77800E-03
300.00	0.74800E-03	400.00	0.71500E-03	500.00	0.67700E-03
600.00	0.64800E-03	700.00	0.61600E-03	800.00	0.58300E-03
900.00	0.55100E-03	1000.0	0.51900E-03	1100.0	0.48400E-03
1200.0	0.45100E-03	1300.0	0.41700E-03	1400.0	0.38000E-03
1500.0	0.36300E-03				

PROPERTY TABLE C MAT= 2 NUM. POINTS= 16

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.10330	100.00	0.10530	200.00	0.11210
300.00	0.11770	400.00	0.12340	500.00	0.12780

600.00	0.13220	700.00	0.13810	800.00	0.14520
900.00	0.15350	1000.0	0.16240	1100.0	0.17100
1200.0	0.18290	1300.0	0.20450	1400.0	0.40100
1500.0	0.19820				

PROPERTY TABLE EX MAT= 3 NUM. POINTS= 6

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.29900E+08	100.00	0.29900E+08	200.00	0.29900E+08
300.00	0.29900E+08	400.00	0.29900E+08	500.00	0.29900E+08

PROPERTY TABLE NUXY MAT= 3 NUM. POINTS= 6

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.30000	100.00	0.30000	200.00	0.30000
300.00	0.30000	400.00	0.30000	500.00	0.30000

PROPERTY TABLE ALPX MAT= 3 NUM. POINTS= 6

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.65000E-05	100.00	0.65000E-05	200.00	0.65000E-05
300.00	0.65000E-05	400.00	0.65000E-05	500.00	0.65000E-05

PROPERTY TABLE DENS MAT= 3 NUM. POINTS= 1

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
0.0000	0.28300				

PROPERTY TABLE KXX MAT= 3 NUM. POINTS= 16

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.81300E-03	100.00	0.80300E-03	200.00	0.77800E-03
300.00	0.74800E-03	400.00	0.71500E-03	500.00	0.67700E-03
600.00	0.64800E-03	700.00	0.61600E-03	800.00	0.58300E-03
900.00	0.55100E-03	1000.0	0.51900E-03	1100.0	0.48400E-03
1200.0	0.45100E-03	1300.0	0.41700E-03	1400.0	0.38000E-03
1500.0	0.36300E-03				

PROPERTY TABLE C MAT= 3 NUM. POINTS= 16

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.10330	100.00	0.10530	200.00	0.11210
300.00	0.11770	400.00	0.12340	500.00	0.12780
600.00	0.13220	700.00	0.13810	800.00	0.14520
900.00	0.15350	1000.0	0.16240	1100.0	0.17100
1200.0	0.18290	1300.0	0.20450	1400.0	0.40100
1500.0	0.19820				

PROPERTY TABLE EX MAT= 4 NUM. POINTS= 6

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.27900E+08	100.00	0.27800E+08	200.00	0.27700E+08
300.00	0.27400E+08	400.00	0.27000E+08	500.00	0.26400E+08

PROPERTY TABLE NUXY MAT= 4 NUM. POINTS= 6

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.30000	100.00	0.30000	200.00	0.30000
300.00	0.30000	400.00	0.30000	500.00	0.30000

PROPERTY TABLE ALPX MAT= 4 NUM. POINTS= 6

TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA
70.000	0.65000E-05	100.00	0.65000E-05	200.00	0.66700E-05
300.00	0.68700E-05	400.00	0.70700E-05	500.00	0.72500E-05



PROPERTY TABLE DENS MAT= 4 NUM. POINTS= 1  
 TEMPERATURE DATA TEMPERATURE DATA TEMPERATURE DATA  
 0.0000 0.28300

PROPERTY TABLE KXX MAT= 4 NUM. POINTS= 16  
 TEMPERATURE DATA TEMPERATURE DATA TEMPERATURE DATA  
 70.000 0.81300E-03 100.00 0.80300E-03 200.00 0.77800E-03  
 300.00 0.74800E-03 400.00 0.71500E-03 500.00 0.67700E-03  
 600.00 0.64800E-03 700.00 0.61600E-03 800.00 0.58300E-03  
 900.00 0.55100E-03 1000.0 0.51900E-03 1100.0 0.48400E-03  
 1200.0 0.45100E-03 1300.0 0.41700E-03 1400.0 0.38000E-03  
 1500.0 0.36300E-03

PROPERTY TABLE C MAT= 4 NUM. POINTS= 16  
 TEMPERATURE DATA TEMPERATURE DATA TEMPERATURE DATA  
 70.000 0.10330 100.00 0.10530 200.00 0.11210  
 300.00 0.11770 400.00 0.12340 500.00 0.12780  
 600.00 0.13220 700.00 0.13810 800.00 0.14520  
 900.00 0.15350 1000.0 0.16240 1100.0 0.17100  
 1200.0 0.18290 1300.0 0.20450 1400.0 0.40100  
 1500.0 0.19820

PROPERTY TABLE EX MAT= 5 NUM. POINTS= 6  
 TEMPERATURE DATA TEMPERATURE DATA TEMPERATURE DATA  
 70.000 0.28300E+08 100.00 0.28100E+08 200.00 0.27600E+08  
 300.00 0.27000E+08 400.00 0.26500E+08 500.00 0.25800E+08

PROPERTY TABLE NUXY MAT= 5 NUM. POINTS= 6  
 TEMPERATURE DATA TEMPERATURE DATA TEMPERATURE DATA  
 70.000 0.30000 100.00 0.30000 200.00 0.30000  
 300.00 0.30000 400.00 0.30000 500.00 0.30000

PROPERTY TABLE ALPX MAT= 5 NUM. POINTS= 6  
 TEMPERATURE DATA TEMPERATURE DATA TEMPERATURE DATA  
 70.000 0.85000E-05 100.00 0.86000E-05 200.00 0.89000E-05  
 300.00 0.92000E-05 400.00 0.95000E-05 500.00 0.97000E-05

PROPERTY TABLE DENS MAT= 5 NUM. POINTS= 1  
 TEMPERATURE DATA TEMPERATURE DATA TEMPERATURE DATA  
 0.0000 0.28300

PROPERTY TABLE KXX MAT= 5 NUM. POINTS= 6  
 TEMPERATURE DATA TEMPERATURE DATA TEMPERATURE DATA  
 70.000 0.19900E-03 100.00 0.20100E-03 200.00 0.21500E-03  
 300.00 0.22700E-03 400.00 0.24100E-03 500.00 0.25200E-03

PROPERTY TABLE C MAT= 5 NUM. POINTS= 6  
 TEMPERATURE DATA TEMPERATURE DATA TEMPERATURE DATA  
 70.000 0.11650 100.00 0.11700 200.00 0.12190  
 300.00 0.12520 400.00 0.12890 500.00 0.13110

PROPERTY TABLE EX MAT= 6 NUM. POINTS= 8  
 TEMPERATURE DATA TEMPERATURE DATA TEMPERATURE DATA  
 -40.000 0.24600E+07 -20.000 0.24300E+07 70.000 0.22700E+07  
 100.00 0.22100E+07 200.00 0.20100E+07 300.00 0.18500E+07  
 400.00 0.17000E+07 500.00 0.15200E+07

PROPERTY TABLE	NUXY	MAT=	6	NUM. POINTS=	6		
TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA		
81.000	0.40000	212.00	0.40000	302.00	0.40000		
392.00	0.40000	513.00	0.40000	621.00	0.40000		
PROPERTY TABLE	ALPX	MAT=	6	NUM. POINTS=	.8		
TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA		
-40.000	0.15560E-04	-20.000	0.15650E-04	70.000	0.16060E-04		
100.00	0.16220E-04	200.00	0.16700E-04	300.00	0.17330E-04		
400.00	0.18160E-04	500.00	0.19120E-04				
PROPERTY TABLE	DENS	MAT=	6	NUM. POINTS=	1		
TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA		
0.0000	0.41000						
PROPERTY TABLE	KXX	MAT=	6	NUM. POINTS=	4		
TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA		
32.000	0.47000E-03	212.00	0.44700E-03	392.00	0.42100E-03		
572.00	0.39800E-03						
PROPERTY TABLE	C	MAT=	6	NUM. POINTS=	5		
TEMPERATURE	DATA	TEMPERATURE	DATA	TEMPERATURE	DATA		
32.000	0.30600E-01	212.00	0.31500E-01	392.00	0.32500E-01		
572.00	0.33500E-01	752.00	0.32800E-01				

Title 8-120B Cask Analyses - Optional Raised Seal Surface on Bolting Ring

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Appendix B

CNS 8-120B Cask Analyses

ANSYS Finite Element Analysis Result Summary

(8 Pages)

CNS 8-120B Cask Analyses  
ANSYS Finite Element Analysis Result Summary

**1. Normal Conditions - Thermal Stresses**

**Primary Lid Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	9331	9331	4301	8777	8777
VALUE	-2225.2	-2657.1	-3662.8	62.395	54.500
MAXIMUM VALUES					
NODE	9275	8859	8859	4301	4301
VALUE	3385.7	1172.6	916.19	3514.3	3082.9

**Seal Plate Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	10460	10441	10453	10613	10613
VALUE	-3028.3	-6411.6	-10382.	986.19	867.17
MAXIMUM VALUES					
NODE	10450	10591	10613	10457	10457
VALUE	3214.0	-602.09	-1270.0	10191.	8869.6

**Bolting Ring Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	10675	11002	11002	11626	11626
VALUE	-939.40	-6016.8	-30747.	1784.3	1546.0
MAXIMUM VALUES					
NODE	10924	10862	10862	10924	10924
VALUE	56878.	21173.	15164.	60116.	52982.

**Inner Shell Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	14391	14239	14199	14201	14201
VALUE	83.882	-2529.0	-5076.5	2467.6	2447.4
MAXIMUM VALUES					
NODE	10689	10693	10686	10689	10689
VALUE	20285.	8892.0	2683.4	18562.	16232.



**Outer Shell Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	10949	12829	10941	13152	13152
VALUE	1343.7	588.02	-4224.4	3440.4	3010.1
MAXIMUM VALUES					
NODE	10871	10868	10871	10870	10870
VALUE	20067.	9415.8	3766.4	16831.	14854.

**Primary Lid Bolt Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	20442	20434	20543	20132	20006
VALUE	-3316.4	-4278.7	-8340.0	512.87	468.39
MAXIMUM VALUES					
NODE	20476	20476	20472	20431	20431
VALUE	13936.	10193.	8420.3	16701.	14949.

**2. Result Summary - Hypothetical End Drop****Primary Lid Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	9363	9449	9363	9226	9226
VALUE	-33821.	-48717.	-88062.	127.45	120.93
MAXIMUM VALUES					
NODE	4301	4271	4271	9665	9665
VALUE	38501.	13208.	12496.	63356.	56988.

**Seal Plate Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	10570	10570	10565	10466	10466
VALUE	-63686.	-67583.	-0.12704E+06	5647.5	5064.3
MAXIMUM VALUES					
NODE	10605	10610	10617	10571	10571
VALUE	54216.	3981.8	-439.02	0.12127E+06	0.10526E+06

**Bolting Ring Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	11515	11515	11516	11010	11010
VALUE	-25683.	-32826.	-55220.	1695.9	1489.3
MAXIMUM VALUES					
NODE	10835	11304	11303	10805	10805
VALUE	30010.	12422.	2292.6	49055.	42487.

**Inner Shell Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	14408	10681	10674	10683	10683
VALUE	-2019.5	-8631.5	-32103.	8439.6	7324.0
MAXIMUM VALUES					
NODE	10690	14258	10683	10674	10674
VALUE	3253.0	74.150	-6227.4	33688.	30153.

**Outer Shell Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	10868	10868	10868	10939	10939
VALUE	-4551.5	-7641.2	-28499.	3372.6	3047.2
MAXIMUM VALUES					
NODE	12829	12831	10937	12886	12886
VALUE	15561.	11768.	102.18	24948.	23755.

**Primary Lid Bolt Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	20457	20584	20547	20107	20107
VALUE	-4230.6	-10462.	-14495.	0.37957	0.33552
MAXIMUM VALUES					
NODE	20599	20563	20469	20563	20563
VALUE	16502.	13060.	4512.3	18668.	17260.

### 3. Hypothetical Corner Drop – Far End

#### Primary Lid Stresses

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	4271	9990	4301	8777	8777
VALUE	-29865.	-54005.	-83358.	776.51	681.18
MAXIMUM VALUES					
NODE	8632	8652	4458	8623	8623
VALUE	0.10072E+06	58220.	22990.	0.12126E+06	0.11227E+06

#### Seal Plate Stresses

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	10485	10458	10458	10519	10519
VALUE	63.853	-5603.2	-26687.	923.59	842.42
MAXIMUM VALUES					
NODE	10592	10448	10394	10458	10458
VALUE	23076.	7810.2	365.56	40353.	34958.

#### Bolting Ring Stresses

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	10836	10806	11420	11613	11657
VALUE	-5319.5	-29138.	-68148.	476.70	427.40
MAXIMUM VALUES					
NODE	12048	12063	10869	11420	11420
VALUE	68046.	14443.	3006.6	88998.	78829.

#### Inner Shell Stresses

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	14237	14329	10663	13795	13795
VALUE	5349.3	-127.30	-3622.5	4303.7	4215.3
MAXIMUM VALUES					
NODE	10683	10674	14184	10685	10686
VALUE	22472.	5986.1	4863.9	22629.	20811.

**Outer Shell Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	12831	12830	12848	13304	13304
VALUE	25.967	-819.64	-5198.2	571.32	495.78
MAXIMUM VALUES					
NODE	10870	10871	12790	10862	10862
VALUE	26612.	5300.7	3155.9	27138.	25062.

**Primary Lid Bolt Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	19985	19990	20002	20285	20285
VALUE	-7413.8	-40804.	-51227.	2290.7	1993.1
MAXIMUM VALUES					
NODE	20180	20182	20179	20356	20329
VALUE	0.22397E+06	91377.	70254.	0.18254E+06	0.17227E+06

**4. Hypothetical Corner Drop – Close End****Primary Lid Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	9915	9915	9955	6297	6297
VALUE	-674.77	-1267.2	-1520.7	4.9068	4.8204
MAXIMUM VALUES					
NODE	10080	10080	10105	10034	10034
VALUE	1253.1	822.01	463.19	1472.0	1276.4

**Seal Plate Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	10616	10456	10458	10645	10471
VALUE	-95.946	-270.14	-1335.2	240.79	215.99
MAXIMUM VALUES					
NODE	10453	10453	10446	10460	10460
VALUE	1991.2	548.14	192.14	2150.8	1865.2



**Bolting Ring Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	10871	10871	10924	11319	11320
VALUE	-3363.8	-4886.2	-13420.	227.86	199.38
MAXIMUM VALUES					
NODE	10680	10677	11359	10924	10924
VALUE	10023.	4794.9	371.73	12620.	11739.

**Inner Shell Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	10690	10686	10688	10698	10698
VALUE	-1540.5	-2219.7	-9008.1	3537.2	3094.5
MAXIMUM VALUES					
NODE	14320	13856	14246	13805	13806
VALUE	13682.	4952.6	0.42798	14512.	13713.

**Outer Shell Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	10873	10871	10868	12917	12917
VALUE	-1418.1	-2072.7	-6676.8	41.649	36.321
MAXIMUM VALUES					
NODE	10941	12829	13210	12886	12886
VALUE	4881.4	2708.1	0.32603	5611.6	5398.4

**Primary Lid Bolt Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	20138	20563	20138	19935	19935
VALUE	-2129.0	-2957.0	-7897.9	159.83	148.96
MAXIMUM VALUES					
NODE	20179	20560	20560	20407	20407
VALUE	9008.3	3292.6	3208.0	8800.8	8422.5

## 5. Hypothetical Fire Accident

### Primary Lid Stresses

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	9915	9982	9982	8780	8780
VALUE	-3171.0	-5593.9	-13315.	22.815	20.580
MAXIMUM VALUES					
NODE	9471	10079	4459	9982	9982
VALUE	4381.0	1850.5	568.55	12181.	10674.

### Seal Plate Stresses

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	10458	10441	10404	10613	10613
VALUE	-4522.0	-11329.	-17786.	4597.7	4019.0
MAXIMUM VALUES					
NODE	10450	10487	10592	10403	10403
VALUE	5316.8	593.69	-5109.7	14908.	13322.

### Bolting Ring Stresses

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	11002	11002	11002	11533	11533
VALUE	-3551.3	-14284.	-62936.	231.40	200.43
MAXIMUM VALUES					
NODE	10924	10862	10862	10924	10924
VALUE	99067.	38172.	24806.	0.10606E+06	93221.

### Inner Shell Stresses

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	14238	10680	14199	14238	14238
VALUE	3262.1	-1429.3	-9605.3	6911.7	6015.2
MAXIMUM VALUES					
NODE	10689	14062	10686	10689	10689
VALUE	59803.	22702.	8222.5	53624.	48627.

**Outer Shell Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	13067	12838	12943	13067	12877
VALUE	5370.2	-4090.4	-7567.4	5571.6	5192.2
MAXIMUM VALUES					
NODE	10871	10868	10871	10870	10870
VALUE	35029.	17510.	6479.6	29072.	25455.

**Primary Lid Bolt Stresses**

	S1	S2	S3	SINT	SEQV
MINIMUM VALUES					
NODE	20469	20434	20138	19935	19935
VALUE	-7124.1	-8829.1	-24793.	535.20	483.28
MAXIMUM VALUES					
NODE	20179	20476	20472	20431	20431
VALUE	29979.	19853.	14512.	28652.	25323.

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Appendix C

CNS 8-120B Cask Analyses  
ANSYS Stress Result Linearization

(10 Pages)



CNS 8-120B Cask Analyses  
ANSYS Stress Result Linearization

1. Normal Conditions - Thermal Stresses

Bolting Ring

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SECTION1 DSYS= 0

\*\*\*\*\* POST1 LINEARIZED STRESS LISTING \*\*\*\*\*  
INSIDE NODE = 10978      OUTSIDE NODE = 11000

LOAD STEP      1    SUBSTEP=      1  
TIME=      1.0000      LOAD CASE=    0

THE FOLLOWING X,Y,Z STRESSES ARE IN COORDINATE SYSTEM 12

```

** MEMBRANE **
      SX      SY      SZ      SXY      SYZ      SXZ
0.1387E+05  6499.    -263.0   -686.4   -1974.   -3240.
      S1      S2      S3      SINT     SEQV
0.1458E+05  7029.    -1508.   0.1609E+05  0.1394E+05

** BENDING **    I=INSIDE C=CENTER O=OUTSIDE
      SX      SY      SZ      SXY      SYZ      SXZ
I  3034.    -2064.    463.5    107.5    66.36    642.9
C  0.000     0.000     0.000     0.000     0.000     0.000
O -3034.     2064.   -463.5   -107.5   -66.36   -642.9
      S1      S2      S3      SINT     SEQV
I  3189.     312.4   -2068.    5257.    4559.
C  0.000     0.000     0.000     0.000     0.000
O  2068.    -312.4   -3189.    5257.    4559.

** MEMBRANE PLUS BENDING **    I=INSIDE C=CENTER O=OUTSIDE
      SX      SY      SZ      SXY      SYZ      SXZ
I  0.1690E+05  4435.    200.5   -578.8   -1908.   -2597.
C  0.1387E+05  6499.   -263.0   -686.4   -1974.   -3240.
O  0.1083E+05  8563.   -726.5   -793.9   -2040.   -3883.
      S1      S2      S3      SINT     SEQV
I  0.1730E+05  5156.   -921.4   0.1822E+05  0.1607E+05
C  0.1458E+05  7029.   -1508.   0.1609E+05  0.1394E+05
O  0.1202E+05  8992.   -2347.   0.1437E+05  0.1312E+05

** PEAK **    I=INSIDE C=CENTER O=OUTSIDE
      SX      SY      SZ      SXY      SYZ      SXZ
I  1.318    -0.8963    0.2013    0.4669E-01  0.2881E-01  0.2792
C  0.3638E-11 -0.2092E-10 -0.8794E-10 -0.6708E-11  0.1592E-11 -0.7321E-10
O -1.318     0.8963   -0.2013   -0.4669E-01 -0.2881E-01 -0.2792
      S1      S2      S3      SINT     SEQV
I  1.385     0.1356   -0.8978    2.282    1.980
C  0.4487E-10 -0.2156E-10 -0.1285E-09  0.1734E-09  0.1515E-09
O  0.8978    -0.1356   -1.385    2.282    1.980

** TOTAL **    I=INSIDE C=CENTER O=OUTSIDE
      SX      SY      SZ      SXY      SYZ      SXZ
I  0.1690E+05  4434.    200.7   -578.8   -1908.   -2597.
    
```

C	0.1387E+05	6499.	-263.0	-686.4	-1974.	-3240.
O	0.1083E+05	8564.	-726.7	-793.9	-2041.	-3883.
	S1	S2	S3	SINT	SEQV	TEMP
I	0.1730E+05	5155.	-921.2	0.1822E+05	0.1607E+05	176.6
C	0.1458E+05	7029.	-1508.	0.1609E+05	0.1394E+05	
O	0.1202E+05	8992.	-2347.	0.1437E+05	0.1312E+05	176.6

2. Hypothetical End Drop

Primary Lid

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SECTION1 DSYS= 0

\*\*\*\*\* POST1 LINEARIZED STRESS LISTING \*\*\*\*\*  
 INSIDE NODE = 9363      OUTSIDE NODE = 8723

LOAD STEP      1    SUBSTEP=      1  
 TIME=      1.0000      LOAD CASE=    0

THE FOLLOWING X,Y,Z STRESSES ARE IN COORDINATE SYSTEM    1

** MEMBRANE **						
	SX	SY	SZ	SXY	SYZ	SXZ
	92.95	-6327.	-0.2645E+05	-70.30	35.31	-1571.
	S1	S2	S3	SINT	SEQV	
	186.4	-6328.	-0.2654E+05	0.2673E+05	0.2414E+05	
** BENDING **    I=INSIDE C=CENTER O=OUTSIDE						
	SX	SY	SZ	SXY	SYZ	SXZ
I	-7787.	-9231.	-0.3076E+05	-122.3	94.96	-3568.
C	0.000	0.000	0.000	0.000	0.000	0.000
O	7787.	9231.	0.3076E+05	122.3	-94.96	3568.
	S1	S2	S3	SINT	SEQV	
I	-7236.	-9240.	-0.3130E+05	0.2407E+05	0.2313E+05	
C	0.000	0.000	0.000	0.000	0.000	
O	0.3130E+05	9240.	7236.	0.2407E+05	0.2313E+05	
** MEMBRANE PLUS BENDING **    I=INSIDE C=CENTER O=OUTSIDE						
	SX	SY	SZ	SXY	SYZ	SXZ
I	-7694.	-0.1556E+05	-0.5721E+05	-192.6	130.3	-5139.
C	92.95	-6327.	-0.2645E+05	-70.30	35.31	-1571.
O	7880.	2904.	4313.	51.98	-59.65	1997.
	S1	S2	S3	SINT	SEQV	
I	-7161.	-0.1556E+05	-0.5774E+05	0.5058E+05	0.4694E+05	
C	186.4	-6328.	-0.2654E+05	0.2673E+05	0.2414E+05	
O	8774.	3429.	2893.	5881.	5632.	
** PEAK **    I=INSIDE C=CENTER O=OUTSIDE						
	SX	SY	SZ	SXY	SYZ	SXZ
I	-0.3285E+05	-0.1827E+05	-0.2955E+05	-7.980	22.25	-2730.
C	3194.	3591.	0.1028E+05	32.15	-18.01	1431.
O	-5109.	-5276.	-0.1517E+05	-49.38	45.87	-1921.
	S1	S2	S3	SINT	SEQV	
I	-0.1827E+05	-0.2801E+05	-0.3439E+05	0.1612E+05	0.1406E+05	
C	0.1056E+05	3593.	2914.	7646.	7330.	

```

O -4748.      -5282.      -0.1553E+05  0.1078E+05  0.1052E+05

      ** TOTAL **  I=INSIDE C=CENTER O=OUTSIDE
      SX          SY          SZ          SXY          SYZ          SXZ
I -0.4055E+05 -0.3383E+05 -0.8676E+05 -200.6      152.5      -7869.
C  3287.      -2735.      -0.1617E+05 -38.15      17.30      -140.2
O  2771.      -2372.      -0.1086E+05  2.596      -13.78      76.02
      S1          S2          S3          SINT          SEQV          TEMP
I -0.3382E+05 -0.3925E+05 -0.8806E+05  0.5424E+05  0.5174E+05  70.00
C  3288.      -2736.      -0.1617E+05  0.1946E+05  0.1725E+05
O  2771.      -2372.      -0.1086E+05  0.1363E+05  0.1192E+05  70.00
    
```

**Bolting Ring**

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SECTION2 DSYS= 0

\*\*\*\*\* POST1 LINEARIZED STRESS LISTING \*\*\*\*\*  
 INSIDE NODE = 10494      OUTSIDE NODE = 10571

LOAD STEP      1    SUBSTEP=      1  
 TIME=      1.0000      LOAD CASE=    0

THE FOLLOWING X,Y,Z STRESSES ARE IN COORDINATE SYSTEM    1

```

      ** MEMBRANE **
      SX          SY          SZ          SXY          SYZ          SXZ
-0.2113E+05 -0.2343E+05 -0.2323E+05 -83.09      21.35      269.3
      S1          S2          S3          SINT          SEQV
-0.2109E+05 -0.2326E+05 -0.2344E+05  2348.      2263.
    
```

```

      ** BENDING **  I=INSIDE C=CENTER O=OUTSIDE
      SX          SY          SZ          SXY          SYZ          SXZ
I  0.4382E+05  0.2428E+05  0.3749E+05 -170.3      -5.415      -3595.
C  0.000      0.000      0.000      0.000      0.000      0.000
O -0.4382E+05 -0.2428E+05 -0.3749E+05  170.3      5.415      3595.
      S1          S2          S3          SINT          SEQV
I  0.4544E+05  0.3586E+05  0.2428E+05  0.2116E+05  0.1835E+05
C  0.000      0.000      0.000      0.000      0.000
O -0.2428E+05 -0.3586E+05 -0.4544E+05  0.2116E+05  0.1835E+05
    
```

```

      ** MEMBRANE PLUS BENDING **  I=INSIDE C=CENTER O=OUTSIDE
      SX          SY          SZ          SXY          SYZ          SXZ
I  0.2269E+05  852.7      0.1426E+05 -253.4      15.93      -3326.
C -0.2113E+05 -0.2343E+05 -0.2323E+05 -83.09      21.35      269.3
O -0.6494E+05 -0.4771E+05 -0.6071E+05  87.24      26.76      3865.
      S1          S2          S3          SINT          SEQV
I  0.2385E+05  0.1311E+05  849.7      0.2300E+05  0.1993E+05
C -0.2109E+05 -0.2326E+05 -0.2344E+05  2348.      2263.
O -0.4771E+05 -0.5842E+05 -0.6723E+05  0.1952E+05  0.1694E+05
    
```

```

      ** PEAK **  I=INSIDE C=CENTER O=OUTSIDE
      SX          SY          SZ          SXY          SYZ          SXZ
I -0.1879E+05 -0.1023E+05 -0.1445E+05  127.5      -69.73      3300.
C  0.1205E+05  5610.      5689.      -126.1      78.40      -2018.
O -1552.      0.3172E+05  0.1095E+06  245.4      -384.0      0.1500E+05
      S1          S2          S3          SINT          SEQV
    
```

```

I -0.1023E+05 -0.1267E+05 -0.2057E+05 0.1034E+05 9364.
C 0.1264E+05 5610. 5100. 7541. 7299.
O 0.1115E+06 0.3172E+05 -3545. 0.1150E+06 0.1020E+06
    
```

```

          ** TOTAL **  I=INSIDE C=CENTER O=OUTSIDE
          SX          SY          SZ          SXY          SYZ          SXZ
I  3897.         -9381.         -189.6         -125.9         -53.79         -25.81
C -9075.         -0.1782E+05 -0.1754E+05 -209.2          99.75         -1749.
O -0.6649E+05 -0.1599E+05 0.4875E+05 332.7          -357.3         0.1886E+05
          S1          S2          S3          SINT          SEQV          TEMP
I  3899.         -189.4          -9382.         0.1328E+05 0.1178E+05 70.00
C -8722.         -0.1779E+05 -0.1792E+05 9197.          9133.
O 0.5176E+05 -0.1599E+05 -0.6951E+05 0.1213E+06 0.1053E+06 70.00
    
```

**Seal Plate**

**3. Hypothetical Corner Drop**

**Primary Lid**

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SECTION1 DSYS= 0

```

***** POST1 LINEARIZED STRESS LISTING *****
INSIDE NODE = 2695    OUTSIDE NODE = 4355
    
```

```

LOAD STEP 1 SUBSTEP= 1
TIME= 1.0000    LOAD CASE= 0
    
```

THE FOLLOWING X,Y,Z STRESSES ARE IN COORDINATE SYSTEM 11

```

          ** MEMBRANE **
          SX          SY          SZ          SXY          SYZ          SXZ
3047.         0.3068E+05 0.1202E+05 -1592.         -573.3         -6395.
          S1          S2          S3          SINT          SEQV
0.3077E+05 0.1534E+05 -369.5         0.3114E+05 0.2697E+05
    
```

```

          ** BENDING **  I=INSIDE C=CENTER O=OUTSIDE
          SX          SY          SZ          SXY          SYZ          SXZ
I 0.2137E+05 0.4533E+05 8347.         -2513.         -631.8         -7925.
C 0.000 0.000 0.000 0.000 0.000 0.000 0.000
O -0.2137E+05 -0.4533E+05 -8347.         2513.         631.8         7925.
          S1          S2          S3          SINT          SEQV
I 0.4559E+05 0.2492E+05 4534.         0.4106E+05 0.3556E+05
C 0.000 0.000 0.000 0.000 0.000 0.000
O -4534.         -0.2492E+05 -0.4559E+05 0.4106E+05 0.3556E+05
    
```

```

          ** MEMBRANE PLUS BENDING **  I=INSIDE C=CENTER O=OUTSIDE
          SX          SY          SZ          SXY          SYZ          SXZ
I 0.2442E+05 0.7601E+05 0.2036E+05 -4106.         -1205.         -0.1432E+05
C 3047.         0.3068E+05 0.1202E+05 -1592.         -573.3         -6395.
O -0.1832E+05 -0.1465E+05 3671.         921.0          58.48         1530.
          S1          S2          S3          SINT          SEQV
I 0.7633E+05 0.3672E+05 7737.         0.6859E+05 0.5964E+05
C 0.3077E+05 0.1534E+05 -369.5         0.3114E+05 0.2697E+05
    
```



```

O  3778.      -0.1444E+05 -0.1864E+05  0.2242E+05  0.2064E+05

      ** PEAK **  I=INSIDE C=CENTER O=OUTSIDE
      SX          SY          SZ          SXY          SYZ          SXZ
I -0.3094E+05   7964.      -0.1175E+05  -1679.      867.3      0.1148E+05
C  5795.      -2391.      5019.      735.5      327.2      1924.
O -4046.      4243.      -2977.      -1025.     -136.0     -1786.
      S1          S2          S3          SINT         SEQV
I  8044.      -6386.     -0.3639E+05  0.4443E+05  0.3926E+05
C  7431.      3452.      -2460.      9890.      8620.
O  4369.      -1688.     -5461.      9830.      8589.

      ** TOTAL **  I=INSIDE C=CENTER O=OUTSIDE
      SX          SY          SZ          SXY          SYZ          SXZ
I -6525.      0.8397E+05  8612.      -5785.     -337.8     -2841.
C  8842.      0.2829E+05  0.1704E+05 -856.8     -246.1     -4472.
O -0.2237E+05 -0.1041E+05  693.9     -103.6     -77.48     -255.9
      S1          S2          S3          SINT         SEQV         TEMP
I  0.8434E+05  9122.      -7403.     0.9174E+05  0.8470E+05  70.00
C  0.2833E+05  0.1900E+05  6838.     0.2149E+05  0.1866E+05
O  697.3      -0.1041E+05 -0.2237E+05  0.2307E+05  0.1998E+05  70.00
    
```

**Bolting Ring**

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SECTION4 DSYS= 0

\*\*\*\*\* POST1 LINEARIZED STRESS LISTING \*\*\*\*\*  
 INSIDE NODE = 20263      OUTSIDE NODE = 20342

LOAD STEP      1    SUBSTEP=      1  
 TIME=      1.0000      LOAD CASE=    0

THE FOLLOWING X,Y,Z STRESSES ARE IN COORDINATE SYSTEM    1

```

      ** MEMBRANE **
      SX          SY          SZ          SXY          SYZ          SXZ
1310.      1417.      0.3526E+05  -105.0      1004.      1845.
      S1          S2          S3          SINT         SEQV
0.3539E+05  1481.      1117.      0.3427E+05  0.3409E+05

      ** BENDING **  I=INSIDE C=CENTER O=OUTSIDE
      SX          SY          SZ          SXY          SYZ          SXZ
I  1319.      2079.     -0.7002E+05  -469.0     -87.12     2945.
C  0.000      0.000      0.000      0.000      0.000      0.000
O -1319.     -2079.     0.7002E+05  469.0      87.12     -2945.
      S1          S2          S3          SINT         SEQV
I  2330.      1190.     -0.7014E+05  0.7247E+05  0.7190E+05
C  0.000      0.000      0.000      0.000      0.000
O  0.7014E+05 -1190.     -2330.      0.7247E+05  0.7190E+05

      ** MEMBRANE PLUS BENDING **  I=INSIDE C=CENTER O=OUTSIDE
      SX          SY          SZ          SXY          SYZ          SXZ
I  2629.      3496.     -0.3476E+05  -573.9     916.7     4790.
C  1310.      1417.      0.3526E+05  -105.0     1004.      1845.
O -8.931     -662.2     0.1053E+06  364.0     1091.     -1100.
      S1          S2          S3          SINT         SEQV
    
```

I	3854.	2900.	-0.3539E+05	0.3924E+05	0.3877E+05
C	0.3539E+05	1481.	1117.	0.3427E+05	0.3409E+05
O	0.1053E+06	150.6	-844.4	0.1061E+06	0.1056E+06

\*\* PEAK \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-2591.	-1587.	3467.	6.177	-1392.	367.8
C	859.4	-273.8	-597.6	158.7	924.9	-913.1
O	-2521.	-4030.	-553.1	873.9	-1424.	1682.
	S1	S2	S3	SINT	SEQV	
I	3845.	-1931.	-2624.	6469.	6152.	
C	1351.	279.1	-1642.	2993.	2627.	
O	569.1	-2356.	-5317.	5886.	5097.	

\*\* TOTAL \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	38.09	1909.	-0.3129E+05	-567.7	-475.7	5158.
C	2169.	1143.	0.3466E+05	53.69	1929.	932.1
O	-2529.	-4693.	0.1047E+06	1238.	-333.2	581.9
	S1	S2	S3	SINT	SEQV	TEMP
I	2213.	565.4	-0.3212E+05	0.3434E+05	0.3354E+05	70.00
C	0.3480E+05	2143.	1032.	0.3377E+05	0.3322E+05	
O	0.1047E+06	-1969.	-5258.	0.1100E+06	0.1084E+06	70.00

**Seal Plate**

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SECTION3 DSYS= 0

\*\*\*\*\* POST1 LINEARIZED STRESS LISTING \*\*\*\*\*  
 INSIDE NODE = 10458 OUTSIDE NODE = 10613

LOAD STEP 1 SUBSTEP= 1  
 TIME= 1.0000 LOAD CASE= 0

THE FOLLOWING X,Y,Z STRESSES ARE IN COORDINATE SYSTEM 1

\*\* MEMBRANE \*\*

	SX	SY	SZ	SXY	SYZ	SXZ
	-6538.	0.1450E+05	-0.1136E+05	1839.	-3565.	5529.
	S1	S2	S3	SINT	SEQV	
	0.1504E+05	-2928.	-0.1551E+05	0.3055E+05	0.2660E+05	

\*\* BENDING \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-4634.	-2965.	-7186.	479.5	-1289.	4025.
C	0.000	0.000	0.000	0.000	0.000	0.000
O	4634.	2965.	7186.	-479.5	1289.	-4025.
	S1	S2	S3	SINT	SEQV	
I	-1569.	-2845.	-0.1037E+05	8800.	8237.	
C	0.000	0.000	0.000	0.000	0.000	
O	0.1037E+05	2845.	1569.	8800.	8237.	

\*\* MEMBRANE PLUS BENDING \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-0.1117E+05	0.1153E+05	-0.1855E+05	2318.	-4853.	9554.

C	-6538.	0.1450E+05	-0.1136E+05	1839.	-3565.	5529.
O	-1904.	0.1746E+05	-4178.	1359.	-2276.	1503.
	S1	S2	S3	SINT	SEQV	
I	0.1233E+05	-4659.	-0.2586E+05	0.3819E+05	0.3314E+05	
C	0.1504E+05	-2928.	-0.1551E+05	0.3055E+05	0.2660E+05	
O	0.1777E+05	-1158.	-5234.	0.2301E+05	0.2126E+05	

\*\* PEAK \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-1832.	1326.	70.07	206.6	-96.39	258.5
C	1830.	-1327.	-73.19	-206.4	95.83	-256.7
O	-1828.	1328.	76.31	206.2	-95.27	255.0
	S1	S2	S3	SINT	SEQV	
I	1344.	100.8	-1882.	3226.	2818.	
C	1879.	-103.5	-1345.	3224.	2817.	
O	1346.	106.2	-1876.	3223.	2816.	

\*\* TOTAL \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-0.1300E+05	0.1286E+05	-0.1848E+05	2525.	-4950.	9813.
C	-4708.	0.1317E+05	-0.1144E+05	1632.	-3469.	5272.
O	-3732.	0.1879E+05	-4102.	1566.	-2372.	1758.
	S1	S2	S3	SINT	SEQV	TEMP
I	0.1367E+05	-5603.	-0.2669E+05	0.4035E+05	0.3496E+05	70.00
C	0.1370E+05	-1822.	-0.1485E+05	0.2855E+05	0.2475E+05	
O	0.1912E+05	-2157.	-6004.	0.2512E+05	0.2344E+05	70.00

**Lid Bolts**

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SECTION2 DSYS= 0

\*\*\*\*\* POST1 LINEARIZED STRESS LISTING \*\*\*\*\*  
 INSIDE NODE = 11421      OUTSIDE NODE = 10836

LOAD STEP      1    SUBSTEP=      1  
 TIME=          1.0000      LOAD CASE=    0

THE FOLLOWING X,Y,Z STRESSES ARE IN COORDINATE SYSTEM 11

\*\* MEMBRANE \*\*

	SX	SY	SZ	SXY	SYZ	SXZ
	1493.	845.6	9852.	-78.11	-2906.	0.1495E+05
	S1	S2	S3	SINT	SEQV	
	0.2147E+05	840.0	-0.1012E+05	0.3158E+05	0.2777E+05	

\*\* BENDING \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	0.1911E+05	0.2378E+05	7476.	974.0	-1908.	9912.
C	0.000	0.000	0.000	0.000	0.000	0.000
O	-0.1911E+05	-0.2378E+05	-7476.	-974.0	1908.	-9912.
	S1	S2	S3	SINT	SEQV	
I	0.2480E+05	0.2397E+05	1594.	0.2321E+05	0.2280E+05	
C	0.000	0.000	0.000	0.000	0.000	
O	-1594.	-0.2397E+05	-0.2480E+05	0.2321E+05	0.2280E+05	

\*\* MEMBRANE PLUS BENDING \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	0.2061E+05	0.2462E+05	0.1733E+05	895.9	-4814.	0.2486E+05
C	1493.	845.6	9852.	-78.11	-2906.	0.1495E+05
O	-0.1762E+05	-0.2293E+05	2376.	-1052.	-998.0	5036.
	S1	S2	S3	SINT	SEQV	
I	0.4424E+05	0.2481E+05	-6498.	0.5074E+05	0.4434E+05	
C	0.2147E+05	840.0	-0.1012E+05	0.3158E+05	0.2777E+05	
O	3628.	-0.1867E+05	-0.2313E+05	0.2676E+05	0.2483E+05	

\*\* PEAK \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-0.3910E+05	6080.	-9432.	4424.	1808.	-0.1949E+05
C	-1091.	-442.2	9049.	-175.3	-492.7	2999.
O	-6619.	-8074.	-9205.	-583.0	29.06	166.7
	S1	S2	S3	SINT	SEQV	
I	6510.	204.5	-0.4917E+05	0.5568E+05	0.5281E+05	
C	9896.	-467.4	-1913.	0.1181E+05	0.1116E+05	
O	-6407.	-8272.	-9220.	2813.	2479.	

\*\* TOTAL \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-0.1849E+05	0.3071E+05	7895.	5319.	-3006.	5376.
C	401.7	403.5	0.1890E+05	-253.4	-3399.	0.1795E+05
O	-0.2424E+05	-0.3101E+05	-6829.	-1635.	-968.9	5203.
	S1	S2	S3	SINT	SEQV	TEMP
I	0.3153E+05	8787.	-0.2021E+05	0.5173E+05	0.4491E+05	70.00
C	0.3015E+05	313.9	-0.1076E+05	0.4091E+05	0.3665E+05	
O	-5320.	-0.2537E+05	-0.3138E+05	0.2606E+05	0.2364E+05	70.00

#### 4. Hypothetical Fire Accident

##### Bolting Ring

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SECTION1 DSYS= 0

\*\*\*\*\* POST1 LINEARIZED STRESS LISTING \*\*\*\*\*  
 INSIDE NODE = 10867      OUTSIDE NODE = 11002

LOAD STEP      1      SUBSTEP=      1  
 TIME=      1.0000      LOAD CASE=      0

THE FOLLOWING X,Y,Z STRESSES ARE IN COORDINATE SYSTEM      1

\*\* MEMBRANE \*\*

	SX	SY	SZ	SXY	SYZ	SXZ
	0.1720E+05	0.1400E+05	-6088.	-2691.	5020.	-0.1504E+05
	S1	S2	S3	SINT	SEQV	
	0.2635E+05	0.1263E+05	-0.1386E+05	0.4021E+05	0.3540E+05	

\*\* BENDING \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	0.7480E+05	0.2450E+05	6537.	3867.	1520.	391.0
C	0.000	0.000	0.000	0.000	0.000	0.000
O	-0.7480E+05	-0.2450E+05	-6537.	-3867.	-1520.	-391.0
	S1	S2	S3	SINT	SEQV	

```

I  0.7510E+05  0.2433E+05  6409.      0.6869E+05  0.6171E+05
C  0.000      0.000      0.000      0.000      0.000
O  -6409.     -0.2433E+05 -0.7510E+05 0.6869E+05 0.6171E+05
    
```

**\*\* MEMBRANE PLUS BENDING \*\* I=INSIDE C=CENTER O=OUTSIDE**

```

      SX      SY      SZ      SXY      SYZ      SXZ
I  0.9200E+05  0.3849E+05  449.0     1176.     6541.     -0.1464E+05
C  0.1720E+05  0.1400E+05 -6088.     -2691.    5020.     -0.1504E+05
O -0.5760E+05 -0.1050E+05 -0.1262E+05 -6558.    3500.     -0.1543E+05
      S1      S2      S3      SINT     SEQV
I  0.9428E+05  0.3956E+05 -2903.     0.9719E+05 0.8439E+05
C  0.2635E+05  0.1263E+05 -0.1386E+05 0.4021E+05 0.3540E+05
O  -3543.     -0.1428E+05 -0.6291E+05 0.5936E+05 0.5479E+05
    
```

**\*\* PEAK \*\* I=INSIDE C=CENTER O=OUTSIDE**

```

      SX      SY      SZ      SXY      SYZ      SXZ
I  32.48     10.64     2.838     1.679     0.6602    0.1698
C  0.1721E-08 0.8949E-09 0.8822E-09 -0.2638E-10 0.9995E-09 -0.1583E-09
O  -32.48     -10.64    -2.838     -1.679    -0.6602   -0.1698
      S1      S2      S3      SINT     SEQV
I  32.61     10.56     2.783     29.83     26.80
C  0.1960E-08 0.1653E-08 -0.1158E-09 0.2076E-08 0.1941E-08
O  -2.783     -10.56    -32.61     29.83     26.80
    
```

**\*\* TOTAL \*\* I=INSIDE C=CENTER O=OUTSIDE**

```

      SX      SY      SZ      SXY      SYZ      SXZ
I  0.9203E+05  0.3851E+05  451.8     1177.     6541.     -0.1464E+05
C  0.1720E+05  0.1400E+05 -6088.     -2691.    5020.     -0.1504E+05
O -0.5763E+05 -0.1051E+05 -0.1263E+05 -6559.    3499.     -0.1543E+05
      S1      S2      S3      SINT     SEQV     TEMP
I  0.9432E+05  0.3957E+05 -2899.     0.9722E+05 0.8441E+05 203.2
C  0.2635E+05  0.1263E+05 -0.1386E+05 0.4021E+05 0.3540E+05
O  -3551.     -0.1428E+05 -0.6294E+05 0.5938E+05 0.5481E+05 201.5
    
```

**Inner Shell**

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SECTION2 DSYS= 0

\*\*\*\*\* POST1 LINEARIZED STRESS LISTING \*\*\*\*\*  
 INSIDE NODE = 10688      OUTSIDE NODE = 10674

LOAD STEP      1    SUBSTEP=      1  
 TIME=      1.0000      LOAD CASE=    0

THE FOLLOWING X,Y,Z STRESSES ARE IN COORDINATE SYSTEM    1

**\*\* MEMBRANE \*\***

```

      SX      SY      SZ      SXY      SYZ      SXZ
8164.     7095.     0.2013E+05  16.43     -112.8     8905.
      S1      S2      S3      SINT     SEQV
0.2488E+05  7095.     3419.     0.2146E+05 0.1988E+05
    
```

**\*\* BENDING \*\* I=INSIDE C=CENTER O=OUTSIDE**

```

      SX      SY      SZ      SXY      SYZ      SXZ
I  2036.     9335.     0.2688E+05  71.35     -50.24     3558.
C  0.000      0.000      0.000      0.000      0.000      0.000
    
```



O	-2036.	-9335.	-0.2688E+05	-71.35	50.24	-3558.
	S1	S2	S3	SINT	SEQV	
I	0.2738E+05	9336.	1535.	0.2585E+05	0.2296E+05	
C	0.000	0.000	0.000	0.000	0.000	
O	-1535.	-9336.	-0.2738E+05	0.2585E+05	0.2296E+05	

\*\* MEMBRANE PLUS BENDING \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	0.1020E+05	0.1643E+05	0.4702E+05	87.78	-163.0	0.1246E+05
C	8164.	7095.	0.2013E+05	16.43	-112.8	8905.
O	6129.	-2241.	-6748.	-54.92	-62.54	5346.
	S1	S2	S3	SINT	SEQV	
I	0.5084E+05	0.1643E+05	6376.	0.4446E+05	0.4039E+05	
C	0.2488E+05	7095.	3419.	0.2146E+05	0.1988E+05	
O	8060.	-2241.	-8679.	0.1674E+05	0.1462E+05	

\*\* PEAK \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-986.2	2517.	9520.	39.32	2.382	-460.6
C	401.1	-861.8	-3337.	-14.17	0.1347E-01	180.0
O	-618.0	929.9	3829.	17.35	-2.436	-259.2
	S1	S2	S3	SINT	SEQV	
I	9540.	2518.	-1007.	0.1055E+05	9300.	
C	409.9	-862.0	-3346.	3756.	3309.	
O	3844.	930.1	-633.3	4477.	3936.	

\*\* TOTAL \*\* I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	9214.	0.1895E+05	0.5654E+05	127.1	-160.6	0.1200E+05
C	8565.	6233.	0.1680E+05	2.263	-112.8	9085.
O	5511.	-1311.	-2919.	-37.57	-64.98	5087.
	S1	S2	S3	SINT	SEQV	TEMP
I	0.5941E+05	0.1895E+05	6341.	0.5307E+05	0.4802E+05	227.5
C	0.2266E+05	6233.	2706.	0.1995E+05	0.1844E+05	
O	7903.	-1311.	-5311.	0.1321E+05	0.1174E+05	234.0

## 4.0 Containment

This chapter describes the containment configuration of the Model CNS 8-120B Package for Normal Transport and Hypothetical Accident Conditions.

### 4.1 Containment Boundary

#### 4.1.1 Containment Vessel

The package containment vessel is defined as the inner shell of the shielded transport cask, together with the associated lid, o-ring seals and lid closure bolts. The inner shell of the cask or containment vessel consists of a right circular cylinder of 62 inches inner diameter and 75 inches inside height. The shell is fabricated of 3/4" thick carbon steel plate, ASTM A516-70. At the base, the cylindrical shell is attached to a circular end plate with full penetration welds. The primary lid is attached to the cask body with twenty (20) equally spaced 2-8 UN bolts. A secondary lid covers an opening in the primary lid and is attached to the primary lid using twelve (12) equally spaced 2-8 UN bolts. See Section 4.1.4 for closure details.

#### 4.1.2 Containment Penetration

There are four penetrations of the containment vessel. These are (1) the optional drain line; (2) the primary lid with the containment boundary of the primary lid's inner o-ring; (3) the secondary lid with the containment boundary of the secondary lid's inner o-ring; and (4) the cask vent port located in the primary lid. Located at the cask base, the drain line consists of a 2" diameter steel rod drilled to 0.75 inches diameter penetrating into the second 3-1/4" layer of steel that forms the cask bottom. A 0.63" DIA. hole, drilled at a right angle, opens on the side of the outer shell near the cask bottom. A vent port penetrates the primary lid into the main cask cavity. The vent and drain penetrations are sealed with Parker Stat-O-Seals or equivalent. The primary and secondary lids are sealed with Parker silicone o-rings or equivalent.

#### 4.1.3 Welds and Seals

The containment vessel is fabricated using full penetration groove welds. Seals are described in Sections 4.1.2 and 4.1.4.

#### 4.1.4 Closure

The primary lid closure consists of two 3-1/4" thick laminated plates, stepped to fit over and within the top edge of the cylindrical body. The lid is supported at the perimeter of the cylindrical body by a thick plate (bolt ring) welded to the top of the inner and outer cylindrical body walls. This plate contains a 14-gauge stainless steel ring at a location, which corresponds to the sealing surface for the o-rings mounted in the lid. The lid is attached to the cask body by twenty (20) equally spaced 2-8 UN bolts. These bolts are torqued to 500 ft-lbs  $\pm$  10 % (lubricated). Two (2) solid, high temperature silicone o-rings are retained in machined grooves at the lid perimeter. Groove dimensions prevent over-compression of the o-rings by the closure bolt pre-load forces and hypothetical accident impact forces. The cask is fitted with a secondary lid of similar construction attached to the primary lid with twelve (12) equally spaced identical bolts. The secondary lid is also sealed with two (2) solid, high temperature silicone o-rings in machined grooves.

The vent and drain penetrations are sealed with Parker Stat-O-Seals (or equivalent), which are used beneath the heads of the hex head cap screws. Table 4.1.4 gives the torque values for the cap screws.

Location	Size (in.)	Torque Values (in-lbs, ± 10% lubricated)	Torque Values (ft-lbs, ± 10% lubricated)
Test Ports (2)	3/8	144	12
Vent	1/2	240	20
Drain	3/4	960	80
Primary Lid	2-8UN	---	500
Secondary Lid	2-8UN	---	500

TABLE 4.1.4 Bolt and Cap Screw Torque Requirements

4.2 Containment Requirements for Normal Conditions of Transport

4.2.1 Leak Test Requirements

The CNS 8-120B cask is designed, fabricated, and leak tested to preclude a release of radioactive material in excess of the limits prescribed in NRC Regulatory Guide 7.4, paragraph C and 10CFR71.51(a)(1). The limits on leakage during normal conditions of transport are defined by 10CFR71.51(a)(1).

The leak test procedure must be able to detect leaks of  $4.94 \times 10^{-6}$  ref-cm<sup>3</sup>/sec (based on dry air at 25°C with a pressure differential of one atmosphere) to assure compliance with 10CFR71.51(a)(1). A description of the calculational procedure used to determine this value follows.

10CFR71.51(a)(1) states the containment requirements for normal conditions of transport as:

*...no loss or dispersal of radioactive contents, as demonstrated to a sensitivity of  $10^{-6}$  A<sub>2</sub> per hour, no significant increase in external radiation levels, and no substantial reduction in the effectiveness of the packaging;*

ANSI N14.5-1997 (Reference 4) states that the permissible leak rate shall be determined by:

$$L := \frac{R}{C}$$

Where:

L = permissible volumetric leak rate (cm<sup>3</sup>/sec)

R = package containment requirements (Ci/sec)

C = activity per unit volume of the medium that could escape from the containment system (Ci/cm<sup>3</sup>)

$$R_N := A_2 \cdot \frac{10^{-6}}{\text{hr}} \quad \Rightarrow \quad R_N = 2.78 \times 10^{-10} \frac{A_2}{\text{sec}}$$

In Section 3.4.4, it is noted that the saturated water vapor in equilibrium at 168 degrees-F and 8.95 psig could exist within the internal shipping containers (liners or drains). It is assumed that these conditions exist within the cask cavity. The containment must limit the leakage of this water vapor to that prescribed in ANSI N14.5. It is very conservative to assume that the concentration of nuclides in the free liquid is equal to that of the solids, which comprise the vast majority of material being transported in the cask. This value is determined below:

Curies = Total Curie Content of Vapor

Void = Minimum Void Volume of Cavity

$$C_N := \frac{\text{Curies}}{V_{\text{Void}}}$$

- Cask curie content is limited to 2000\*A<sub>2</sub> or less (Chapter 1).
- Free water is limited to the restriction of one-percent of solid volume

$$\Rightarrow \text{Curies} := 1\% \cdot 2000 \cdot A_2$$

Vapor Curie Content

$$\Rightarrow \text{Curies} = 20A_2$$



The minimum void volume occurs when the largest liner is shipped.

$$V_{\text{Cavity}} := \frac{\pi \cdot 61.8^2 \cdot 74.9}{4} \cdot \text{in}^3 \quad \text{See Chapter 1 for Cask Dimensions}$$

$$\Rightarrow V_{\text{Cavity}} = 224672 \text{in}^3$$

$$V_{\text{Liner}} := \frac{\pi \cdot 60^2 \cdot 73}{4} \cdot \text{in}^3 \quad \Rightarrow \quad V_{\text{Liner}} = 206403 \text{in}^3$$

$$V_{\text{Void}} := V_{\text{Cavity}} - V_{\text{Liner}} \quad \Rightarrow \quad V_{\text{Void}} = 18269 \text{in}^3$$

$$\Rightarrow \quad V_{\text{Void}} = 299379 \text{cm}^3$$

$$C_{\text{N}} := \frac{\text{Curies}}{V_{\text{Void}}} \quad \Rightarrow \quad C_{\text{N}} = 6.68 \times 10^{-5} \frac{\text{A}_2 \cdot \text{C}_i}{\text{cm}^3}$$

$$L_{\text{N}} := \frac{R_{\text{N}}}{C_{\text{N}}}$$

$$L_{\text{N}} = 4.16 \times 10^{-6} \frac{\text{cm}^3}{\text{sec}}$$

A leak rate at standard conditions will be calculated which is equivalent to a volumetric leak rate of  $4.16 \times 10^{-6} \text{ cm}^3/\text{sec}$  using Equations B.3, B.4, and B.5 from ANSI N14.5-1997 (Ref. 4).

First, determine the diameter of the leakage hole,  $D_{\text{max}}$ , that would allow a leakage of  $4.16 \times 10^{-6} \text{ cm}^3/\text{sec}$ .

$$M_{\text{air}} := 29 \cdot \frac{\text{gm}}{\text{mole}}$$

$$T := 350 \cdot \text{K}$$

$$L_{\text{u}} := (F_{\text{c}} + F_{\text{m}}) \cdot (P_{\text{u}} - P_{\text{d}}) \cdot \left( \frac{P_{\text{a}}}{P_{\text{u}}} \right) \cdot \frac{\text{cm}^3}{\text{sec}} \quad \text{Equation B.5, ANSI N14.5-1997}$$

$$F_m := \frac{3.8 \cdot 10^3 \cdot D^3 \cdot \sqrt{\frac{T}{M_{\text{air}}}}}{a \cdot P_a}$$

Equation B.4, ANSI N14.5-1997

$$F_c := \frac{2.49 \cdot 10^6 \cdot D^4}{a \cdot \mu_{\text{air}}}$$

Where:

$L_u$  = upstream leakage rate,  $\text{cm}^3/\text{sec}$

$$\mu_{\text{air}} := 0.0185 \cdot \text{cP}$$

$$T := 350 \cdot \text{K} \quad \text{Section 3.4.4}$$

$$P_u := 23.65 \cdot \text{psi}$$

$$P_u = 8.95 \text{ psig} = 1.61 \text{ atm}$$

$$P_u = 1.61 \text{ atm}$$

$$P_d := 1 \cdot \text{atm}$$

$$P_a := \frac{P_u + P_d}{2}$$

$$P_a = 1.30 \text{ atm}$$

$$M_{\text{air}} := 29 \cdot \frac{\text{gm}}{\text{mole}}$$

$$a := 0.6 \cdot \text{cm} \quad \text{assumption for length of hole leaking the air}$$

Substitute these parameters into equations B.3, B.4, and B.5 respectively:

$$F_m(D) := \frac{3.81 \cdot 10^3 \cdot D^3 \cdot \sqrt{\frac{350}{29}}}{(0.6) \cdot (1.30)} \quad F_m(D) = 1.70 \times 10^4 D^3$$

$$F_c(D) := \frac{2.49 \cdot 10^6 \cdot D^4}{(.6) \cdot (.0185)} \quad F_c(D) = 2.24 \times 10^8 D^4$$

$$F(D) := \left[ \left[ (2.24 \cdot 10^8 \cdot D^4) + (1.70 \cdot 10^4 \cdot D^3) \right] \cdot (1.61 - 1) \cdot \frac{1.30}{1.61} \right] - 4.16 \cdot 10^{-6}$$

Solve this equation iteratively.

$$D := 4.23 \cdot 10^{-4} \cdot \text{cm}$$

Next, using Equation B.5 of Ref. 4, determine the flow of air at standard conditions through a hole of this diameter:

$$a := 0.6 \cdot \text{cm}$$

$$M_{\text{air}} := 29 \cdot \frac{\text{gm}}{\text{mole}}$$

$$\mu_{\text{air}} := 0.0185 \cdot \text{cP}$$

$$P_u := 1.0 \cdot \text{atm}$$

$$P_d := 0.1 \cdot \text{atm}$$

$$P_a := \frac{P_u + P_d}{2} \quad P_a = 0.55 \text{atm}$$

$$T := 298 \cdot \text{K}$$

$$F_m := \frac{3.81 \cdot 10^3 \cdot D^3 \cdot \sqrt{\frac{T}{M_{\text{air}}}} \cdot \text{cm} \cdot \text{gm}^{.5}}{a \cdot P_a \cdot \text{mole}^{.5} \cdot \text{sec} \cdot \text{K}^{.5}} \quad F_m = 2.801 \times 10^{-6} \frac{\text{cm}^3}{\text{atm} \cdot \text{sec}}$$

$$F_c := \frac{2.49 \cdot 10^6 \cdot D^4 \cdot \text{cP}}{a \cdot \mu_{\text{air}} \cdot \text{sec} \cdot \text{atm}} \quad F_c = 7.182 \times 10^{-6} \frac{\text{cm}^3}{\text{sec} \cdot \text{atm}}$$

Substitute these values into Equation B.5 of Ref. 4:

$$L_{\text{STD}} := (F_c + F_m) \cdot (P_u - P_d) \cdot \frac{P_a}{P_u}$$

$$L_{\text{STD}} = 4.94 \times 10^{-6} \frac{\text{ref cm}^3}{\text{sec}}$$

#### 4.2.2 Pressurization of the Containment Vessel

Section 2.4.4 summarizes normal condition temperatures and pressures within the containment vessel. These pressures and associated temperatures are used to evaluate the integrity of the CNS 8-120B package. None of these conditions reduce the effectiveness of the package containment.

#### 4.2.3 Coolant Containment

Not applicable; there are no coolants in the CNS 8-120B package.

#### 4.2.4 Coolant Loss

Not applicable; there are no coolants in the CNS 8-120B package.

### 4.3 Containment Requirements for the Hypothetical Accident Conditions

#### 4.3.1 Fission Gas Products

There are no fission gas products present.

#### 4.3.2 Leak Test Requirements

Section 2.7 demonstrates that the CNS 8-120B cask will maintain its containment capability throughout the hypothetical accident conditions. Fission gas products will not be carried within the cask so there can be no release of fission gases. The CNS 8-120B cask is designed, fabricated, and leak tested to preclude a release of radioactive material in excess of the limits prescribed in NRC Regulatory Guide 7.4, paragraph C and 10CFR71.51(a)(2). The limits on leakage during hypothetical accident conditions are defined by 10CFR71.51(a)(2).

The leak test procedure which assures compliance with leakage during normal conditions of

transport will also be sufficient to assure compliance during hypothetical accident conditions. A description follows of the calculational procedure which demonstrates that the maximum leakage requirement during normal conditions of transport is more stringent than the maximum leakage requirement during the hypothetical accident.

10CFR71.51(a)(2) states the containment requirements for the hypothetical accident conditions as:

...no escape of krypton-85 exceeding 10,000 curies in one week, no escape of other radioactive material exceeding a total amount of  $A_2$  in one week, and no external radiation dose rate exceeding one rem per hour at one meter from the external surface of the package.

Since the cask does not carry fission products or radioactive gases, only the  $A_2$  per week requirement is limiting. A release of  $A_2$  in one week is equivalent to the activity release rate,  $R_a$ , given by the following equation.

$$\begin{aligned} R_a &= (A_2/\text{week})(1 \text{ week}/168 \text{ hr}) \\ &= 5.952 \times 10^{-3} A_2 / \text{hr} \end{aligned}$$

Next, determine  $L_A$  for hypothetical accident conditions:

$$R_A := 1.65 \cdot 10^{-6} \cdot \frac{A_2}{\text{sec}}$$

$$C_A := C_N \quad \text{Section 4.2.1}$$

$$L_A := \frac{R_A}{C_A} \quad \Rightarrow \quad L_A = 0.025 \frac{\text{cm}^3}{\text{sec}}$$

Since  $L_N$  is more restrictive than  $L_A$ , the standard leak rate is calculated in Section 4.2.1 based on the permissible leak rate of  $L_N = 4.94 \times 10^{-6} \text{ cm}^3/\text{sec}$ .

#### 4.4 Determination of Test Conditions for Assembly Verification Leak Test

This test will be performed by pressurizing the annulus between the o-ring seals of the primary and secondary lids with air using a pressure gauge, readable without estimation, and calibrated to a maximum error of 1% of full scale.

The test pressure shall be 18 psig and the test shall last for 1 hour with an allowable pressure drop of 1 psi. Any condition which results in a pressure drop of more than 1 psi shall be corrected.

The following sections demonstrate that the sensitivity at the test conditions is equivalent to the required procedure sensitivity of  $5 \times 10^{-4} \frac{\text{atm} \cdot \text{cm}^3}{\text{s}}$  based on dry air at standard conditions as defined in ANSI N14.5-1977.

##### 4.4.1 Maximum Permissible Leak Rates at Standard Conditions

The sensitivity of the assembly verification leak test must demonstrate that not more than an A<sub>2</sub> quantity of the radioactive contents is released in 10 days. The sensitivity of the leakage test need not be more sensitive than  $1 \times 10^{-3} \frac{\text{atm} \cdot \text{cm}^3}{\text{s}}$  of dry air at standard conditions.

Furthermore, ANSI N14.5 requires that the leakage test procedure have a sensitivity of one-half the maximum permissible leakage rate, unless otherwise justified. Thus, a procedure sensitivity of  $5 \times 10^{-4} \frac{\text{atm} \cdot \text{cm}^3}{\text{s}}$  of dry air at standard conditions is required.

##### 4.4.2 Detector Sensitivity – Test Conditions

This test will be performed by the pressure drop test described in Section A3.1 of ANSI N14.5 – 1977 by pressurizing the annulus between the o-ring seals with air. The annulus is a ¼” deep groove, 1/8” in width, centered between the o-rings, with an inner diameter of 64.0 inches. The



volume of the annulus is 103.2 cm<sup>3</sup>. The leakage rate at test conditions determined from Equation B.11 of ANSI N14.5-1977 as given below in slightly different terms:

$$L_t = \frac{VT_s \Delta P}{T_t} \left( \frac{\text{atm} \cdot \text{cm}^3}{\text{s}} \right) \quad (1)$$

where:

$L_t$  = equivalent leakage rate of air at the test pressure converted to the standard temperature of 298K.

$V$  = test volume (cm<sup>3</sup>)

$T_s$  = 298K

$\Delta P$  = allowable change in pressure (atm)

$T$  = test temperature (K)

$t$  = test time (s)

#### 4.4.3 Equivalent Leakage Rate at Standard Conditions

Leakage rates are normally expressed as an equivalent leakage rate of dry air at the standard conditions of 298K (25°C) and a 1 atm pressure against a vacuum of 0.1 atm or less. In the case of laminar flow, leakage rates at test conditions, with upstream and downstream pressures different from standard conditions, are proportional to the equivalent leakage rate at standard conditions in the manner described by Equation B.5 of ANSI N14.5-1977 as given below:

$$L_s = \frac{L_t n_t}{n_s [(P_u)^2 - (P_o)^2]} \frac{\text{atm} \cdot \text{cm}^3}{\text{s}} \quad (2)$$

where:

$L_s$  = equivalent leakage rate of dry air at standard conditions.

$n_s = 0.0184$  cP

$L_t$  = equivalent leakage rate of air at the test pressure converted to the standard temperature of 298K

$n_t$  = viscosity of air at test conditions (cP).

$P_u$  = upstream test pressure (atm).

$P_o$  = downstream test pressure (atm).

Substituting equation (1) into equation (2) yields:

$$L_s = \frac{VT_s n_t P}{T t n_s [(P_u)^2 - (P_o)^2]} \quad \frac{\text{atm} \cdot \text{cm}^3}{\text{s}} \quad (3)$$

Where each of the above terms has been described previously.

#### 4.4.4 Required Charge Pressure at Test Temperature

In the specific case of performing a pressure drop leak test on the 8-120B cask the annulus between the o-ring seals will be pressurized with air and the allowable change in pressure will be fixed at 1 psi (0.068 atm). The downstream test pressure in this instance is 1 atm and the test time will be fixed at 1 hour (3600 sec). Insertion of the various constants into equation (3) reduces it to:

$$L_s = \frac{31.43 n_t}{[(P_t)^2 - 1] \Gamma} \quad \frac{\text{atm} \cdot \text{cm}^3}{\text{s}} \quad (4)$$

Where:  $P_t = P_u$ , the upstream test pressure (atm).

In the above equation “ $n_t$ ” is a function of the test temperature. Since the variation in the test air viscosity with the test temperature is nearly linear over the temperature range for which the cask will be leak tested, the test air viscosity will be interpolated between  $n = 0.01708$  cP at 273 K (32°F) and  $N = 0.02102$  cP at 347 K (165°F) (ref. 1, p. F-47).

Functionally this is expressed as:

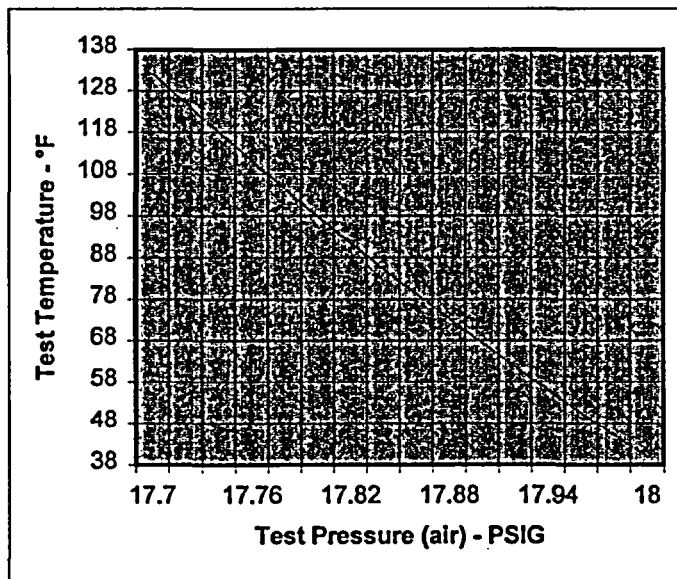
$$n_t = 5.324 \times 10^{-5} T + 2.545 \times 10^{-3} \quad (\text{cP}) \quad (5)$$

Substitution of equation (5) into equation (4) yields:

$$L_s = \frac{1.673 \times 10^{-3} T + 0.080}{[(P_t)^2 - 1] \Gamma} \frac{\text{atm} \cdot \text{cm}^3}{\text{s}} \quad (6)$$

When “ $L_s$ ” of equation (6) is equated to the required procedure sensitivity for leaktightness of  $5 \times 10^{-4} \frac{\text{atm} \cdot \text{cm}^3}{\text{s}}$  of dry air at standard conditions then a functional relationship between the test temperature  $T$  and the required test pressure remains.

This relationship is shown on the figure below with temperature converted to a Fahrenheit scale and pressure converted to a psig scale. From the figure, it is clear that a charge pressure of 18 psig (2.22 atm) is sufficient for leak testing the cask at temperatures above 40°F (278K).



Allowable Pressure Drop = 1 psi

Test Time = 1 Hour (3600 s)

$L_s = 5.0 \times 10^{-4} \text{ atm} \cdot \text{cm}^3/\text{s}$  (dry air & stand. conditions)

#### 4.5 Periodic Verification Leak Rate Determination Using R-12 Test Gas

##### **Calculation of Permitted R-12 Leak Rate**

The purpose of this calculation is to determine the allowable leak rate using the R-12 halogen gas that may be used to perform the annual verification leak tests on the CNS 8-120B cask.

The text of this document is prepared using Mathcad, Version 2000i, software. Most conventions used in the text are the same as normal practice. A benefit of the Mathcad code is that it automatically carries all units with the variables used in the calculations. The code also allows output of variables in any form of the fundamental units (length, mass, time, etc.), allowing for automatic conversions between unit systems without the need for conversion factors. As a courtesy, several conversion factors are presented within the text, and the important final calculated values are listed in tabular form at the end of this document for use in checking the calculations, so that the calculations can be documented as hand calculations, eliminating the need for full verification of the Mathcad software.

This calculation uses formulas presented in ANSI N14.5 – 1997 (Ref. 4).

This test is performed using an R-12 leak detector. A leak standard, traceable to NIST, is used to calibrate the leak detector to detect the maximum allowable leak rates specified in Figure 4.3.

First, using the Standard Allowable Leak Rate from Section 4.3.2 of the SAR, determine the maximum possible diameter hole in the cask o-ring that would permit this leak rate at standard conditions.

$$L_{\text{std}} := 4.94 \cdot 10^{-6} \cdot \text{ref} \cdot \frac{\text{cm}^3}{\text{sec}}$$

See Section 4.2.1

Where, for air at standard conditions:

$$P_a := .505 \cdot \text{atm}$$

$$T := 298 \cdot \text{K}$$

$$P_u := 1.0 \cdot \text{atm}$$

$$M_{\text{air}} := 29 \cdot \frac{\text{gm}}{\text{mole}}$$

$$P_d := .01 \cdot \text{atm}$$

$$\mu_{\text{air}} := 0.0185 \cdot \text{cP}$$

Assume  $a$ , which is the length of the hole in the o-ring, is:

$$a := 0.6 \cdot \text{cm}$$

$$D_{\text{max}} := 4.23 \cdot 10^{-4} \cdot \text{cm} \quad \text{From Section 4.2.1}$$

$$F_c(D) := \frac{2.49 \cdot 10^6 \cdot D^4 \cdot \text{cP} \cdot \text{ref}}{a \cdot \mu_{\text{air}} \cdot \text{sec} \cdot \text{atm}} \quad \text{Eqn. B3 - ANSI N14.5}$$

$$F_m(D) := \frac{3.81 \cdot 10^3 \cdot D^3 \cdot \sqrt{\frac{T}{M_{\text{air}}} \cdot \text{cm} \cdot \text{gm}^{0.5}}}{a \cdot P_a \cdot \text{K}^{0.5} \cdot \text{mole}^{0.5} \cdot \text{sec}} \quad \text{Eqn. B4 - ANSI N14.5}$$

$$L_{\text{std}}(D) := (F_c(D) + F_m(D)) \cdot (P_u - P_d) \cdot \frac{P_a}{P_u} \quad \text{Eqn. B5 - ANSI N14.5}$$

Next, determine the equivalent air/R12 mixture ( $L_{\text{mix}}$ ) that would leak from  $D_{\text{max}}$  during a leak test. Assume the O-ring void is first evacuated to 20"Hg vacuum (9.92"Hg abs) and then pressurized to 25 psig (2.7 atm) with an air/R12 mixture.

$$P_{\text{mix}} := 2.7 \cdot \text{atm}$$

$$P_{\text{air}} := 9.92 \cdot \text{in\_Hg} \quad P_{\text{air}} = 0.33 \text{atm}$$

$$P_{\text{R12}} := P_{\text{mix}} - P_{\text{air}} \quad P_{\text{R12}} = 2.37 \text{atm}$$

$$P_d := 1.0 \cdot \text{atm}$$

$$P_a := \frac{P_{\text{mix}} + P_d}{2} \quad P_a = 1.85 \text{atm}$$

$$M_{\text{R12}} := 121 \cdot \frac{\text{gm}}{\text{mole}} \quad \text{ANSI N14.5 - 1997}$$

$$\mu_{\text{R12}} := 0.0124 \cdot \text{cP} \quad \text{ANSI N14.5 - 1997}$$

$$M_{\text{mix}} := \frac{M_{\text{R12}} \cdot P_{\text{R12}} + M_{\text{air}} \cdot P_{\text{air}}}{P_{\text{mix}}} \quad \text{Eqn. B.7 - ANSI N14.5}$$

$$M_{\text{mix}} = 109.7 \frac{\text{gm}}{\text{mole}}$$

$$\mu_{\text{mix}} := \frac{\mu_{\text{air}} \cdot P_{\text{air}} + \mu_{\text{R12}} \cdot P_{\text{R12}}}{P_{\text{mix}}} \quad \text{Eqn. B.8 - ANSI N14.5}$$

$$\mu_{\text{mix}} = 0.013 \text{cP}$$

Determine  $L_{\text{mix}}$  as a function of temperature. Assume the viscosities of air and R12 do not change significantly over the range of temperatures evaluated:



$$T := 273\text{-K}, 278\text{-K}.. 318\text{-K}$$

Temperature range for test: 32°F to 113°F

$$F_c := \frac{2.49 \cdot 10^6 \cdot D_{\max}^4 \cdot cP \cdot \text{ref}}{a \cdot \mu_{\text{mix}} \text{sec} \cdot \text{atm}}$$

then,

$$F_c = 1.01 \times 10^{-5} \frac{\text{cm}^3}{\text{sec} \cdot \text{atm}}$$

$$F_m(T) := \frac{3.81 \cdot 10^3 \cdot D_{\max}^3 \cdot \sqrt{\frac{T}{M_{\text{mix}}}} \cdot \text{cm} \cdot \text{gm}^{0.5}}{a \cdot P_a \cdot K^{0.5} \cdot \text{mole}^{0.5} \cdot \text{sec}}$$

$$L_{\text{mix}}(T) := \left[ (F_c + F_m(T)) \cdot (P_{\text{mix}} - P_d) \cdot \frac{P_a}{P_{\text{mix}}} \right]$$

$$T_F(T) := \left[ (T \cdot F - 273 \cdot K) \cdot \frac{9}{5 \cdot K} + 32 \right]$$

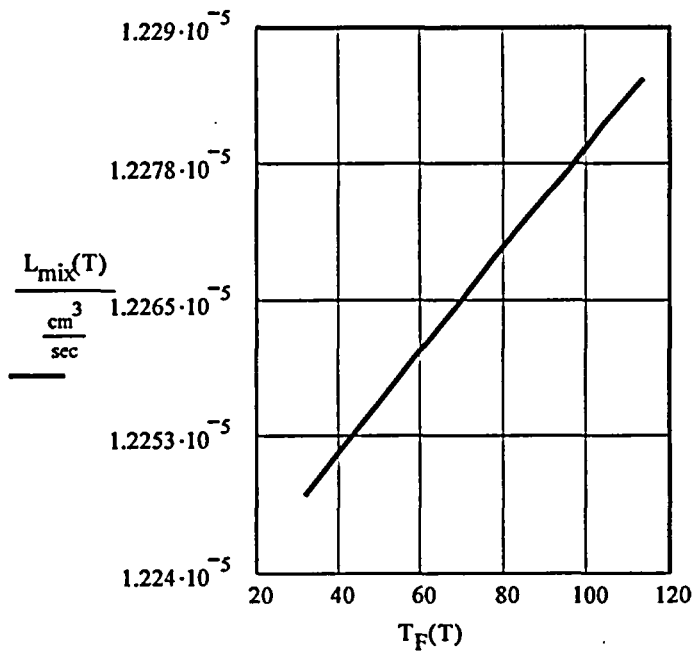


Fig. 4.1 - Allowable R-12/Air Mixture Test Leakage,  $cm^3/sec$ , versus test temperature, deg.F

The R-12 component of this leak rate can be determined by multiplying the leak rate of the mixture by the ratio of the R-12 partial pressure to the total pressure of the mix, as follows.

$$L_{R12}(T) := L_{mix}(T) \cdot \frac{P_{R12}}{P_{mix}}$$

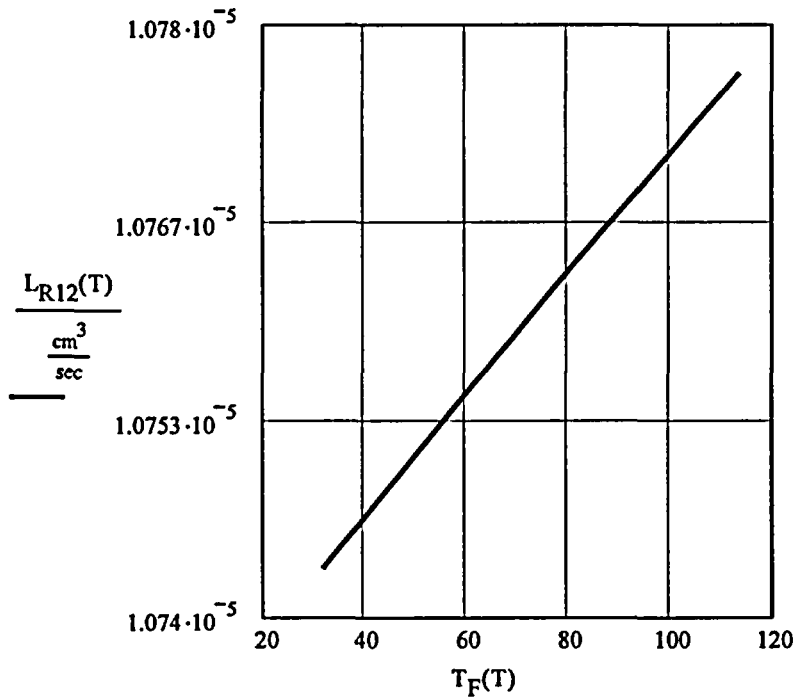


Fig. 4.2 - Allowable R-12 test leakage, cm<sup>3</sup>/sec versus test temperature, deg.F

Determine the equivalent mass flow rate for  $L_{R12}$  in oz/yr:

$$N(T) := \frac{P_{R12} \cdot V}{R_0 \cdot T} \quad \text{Ideal Gas Law}$$

where,

$$R_0 := \frac{82.05 \text{ cm}^3 \cdot \text{atm}}{\text{mole} \cdot \text{K}}$$

This data can then be used to convert the volumetric leak rate for R-12 calculated above to a mass leak rate. By dividing  $N$  by  $V$ , the number of moles per unit volume can be multiplied by the molecular weight of the gas and the maximum allowable volumetric leak rate to determine the maximum allowable mass leak rate, as a function of test temperature as shown in the graph below. The conversion from grams per second to ounces per year is also shown below.

$$L(T) := L_{R12}(T) \cdot \frac{N(T)}{V} \cdot M_{R12} \cdot \frac{yr}{oz}$$

$$\frac{gm}{sec} = 1.113 \times 10^6 \frac{oz}{yr} \quad \text{Conversion of gm/sec to oz/yr}$$

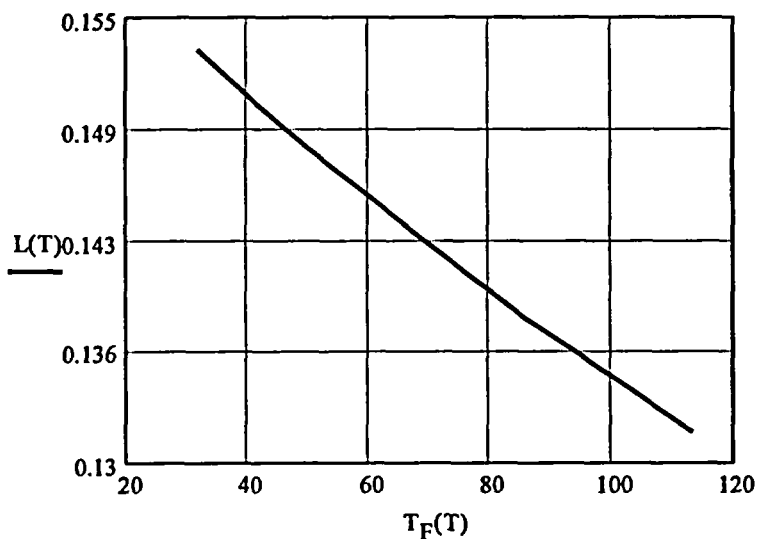


Fig. 4.3 - Allowable R12 test leakage, oz/yr, versus test temperature, deg.F

The graph above can be used to determine the allowable leak rate based on the temperature at the time of the test. According to ANSI N14.5 methodology, the maximum allowable leak rate must be divided by 2 to determine the minimum sensitivity for the test. A graph of the required sensitivity in oz/yr is presented below:

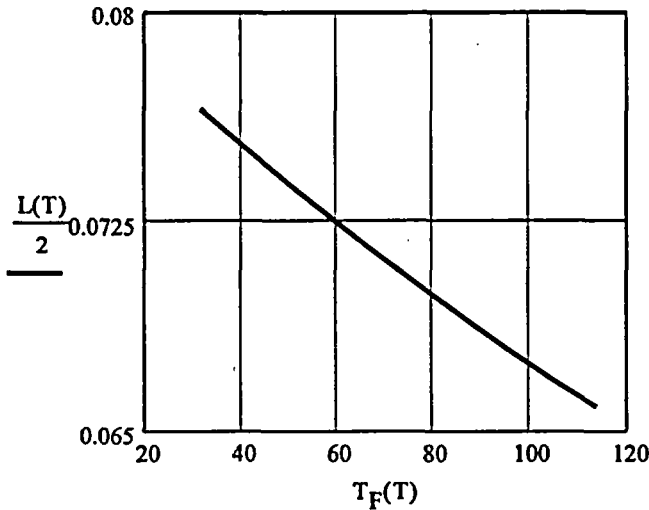


Fig. 4.4 - Allowable R12 test leakage sensitivity, oz/yr, versus test temperature, deg.F

The values presented in Figure 4.4 should be used to determine the sensitivity to calibrate the leak detector prior to the test.

#### 4.6 Periodic Verification Leak Rate Determination Using R-134A Test Gas

##### **Calculation of Permitted R-134a Leak Rate**

The purpose of this calculation is to determine the allowable leak rate using the R-134a halogen gas that will be used as an alternative to perform the annual verification leak tests on the CNS 8-120B cask. This halogen gas is now in widespread use as a replacement gas for R-12 in many industrial applications. Properties for R134a are attached in Appendix 4.1.

The text of this document is prepared using Mathcad, Version 2001i, software. Most conventions used in the text are the same as normal practice. A benefit of the Mathcad code is that it automatically carries all units with the variables used in the calculations. The code also allows output of variables in any form of the fundamental units (length, mass, time, etc.), allowing for automatic conversions between unit systems without the need for conversion factors. As a courtesy, several conversion factors are presented within the text, and the important final calculated values are listed in tabular form at the end of this document for use in checking the calculations,

so that the calculations can be documented as hand calculations, eliminating the need for full verification of the Mathcad software.

This calculation uses formulas presented in ANSI N14.5 - 1997.

This test is performed using an R-134a leak detector. A leak standard, traceable to NIST, is used to calibrate the leak detector to detect the maximum allowable leak rates specified in Figure 4.7.

First, using the Standard Allowable Leak Rate from Section 4.3.2 of the SAR, determine the maximum possible diameter hole in the cask o-ring that would permit this leak rate at standard conditions.

$$L_{std} := 4.94 \cdot 10^{-6} \cdot \text{ref} \cdot \frac{\text{cm}^3}{\text{sec}} \quad \text{See Section 4.2.1.}$$

Where, for air at standard conditions:

$$P_a := .505 \cdot \text{atm}$$

$$T := 298 \cdot \text{K}$$

$$P_u := 1.0 \cdot \text{atm}$$

$$M_{air} := 29 \cdot \frac{\text{gm}}{\text{mole}}$$

$$P_d := .01 \cdot \text{atm}$$

$$\mu_{air} := 0.0185 \text{ cP}$$

Assume  $a$ , which is the length of the hole in the o-ring is:

$$a := 0.6 \cdot \text{cm}$$

The maximum possible diameter of hole in the o-ring is:

$$D_{\max} := 4.23 \cdot 10^{-4} \cdot \text{cm} \quad \text{From 4.2.1}$$

$$F_c(D) := \frac{2.49 \cdot 10^6 \cdot D^4 \cdot cP \cdot \text{ref}}{a \cdot \mu_{\text{air}} \cdot \text{sec} \cdot \text{atm}} \quad \text{Eqn. B3 - ANSI N14.5}$$

$$F_m(D) := \frac{3.81 \cdot 10^3 \cdot D^3 \cdot \sqrt{\frac{T}{M_{\text{air}}}} \cdot \text{cm} \cdot \text{gm}^{0.5}}{a \cdot P_a \cdot K^{0.5} \cdot \text{mole}^{0.5} \cdot \text{sec}} \quad \text{Eqn. B4 - ANSI N14.5}$$

$$L_{\text{std}}(D) := (F_c(D) + F_m(D)) \cdot (P_u - P_d) \cdot \frac{P_a}{P_u} \quad \text{Eqn. B5 - ANSI N14.5}$$

Next, determine the equivalent air/R134a mixture ( $L_{\text{mix}}$ ) that would leak from  $D_{\max}$  during a leak test. Assume the O-ring void is first evacuated to 20"Hg vacuum (9.92"Hg abs) and then pressurized to 25 psig (2.7 atm) with an air/R134a mixture.

$$P_{\text{mix}} := 2.7 \cdot \text{atm}$$

$$P_{\text{air}} := 9.92 \cdot \text{in\_Hg}$$

$$P_{\text{air}} = 0.33 \text{atm}$$



$$P_{R134a} := P_{mix} - P_{air}$$

$$P_{R134a} = 2.37 \text{ atm}$$

$$P_d := 1 \cdot \text{atm}$$

$$P_a := \frac{P_{mix} + P_d}{2}$$

$$P_a = 1.85 \text{ atm}$$

The properties of R134a are given in the attached literature:

$$M_{R134a} := 102 \frac{\text{gm}}{\text{mole}}$$

$$\mu_{R134a} := 0.012 \text{ cP}$$

$$M_{mix} := \frac{M_{R134a} \cdot P_{R134a} + M_{air} \cdot P_{air}}{P_{mix}}$$

$$M_{mix} = 93.04 \frac{\text{gm}}{\text{mole}}$$

$$\mu_{mix} := \frac{\mu_{air} \cdot P_{air} + \mu_{R134a} \cdot P_{R134a}}{P_{mix}}$$

Eqn. B.7 - ANSI N14.5

$$\mu_{mix} = 0.013 \text{ cP}$$

Eqn. B.8 - ANSI N14.5

Determine  $L_{mix}$  as a function of temperature. Assume the viscosities of air and R134a do not change significantly over the range of temperatures evaluated:

$T := 273\text{-K}, 278\text{-K}.. 318\text{-K}$     Temperature range for test: 32°F to 113°F

$$F_c := \frac{2.49 \cdot 10^6 \cdot D_{\max}^4 \cdot cP \cdot \text{ref}}{a \cdot \mu_{\text{mix}} \cdot \text{sec} \cdot \text{atm}}$$

then,

$$F_c = 1.038 \times 10^{-5} \frac{\text{cm}^3}{\text{sec} \cdot \text{atm}}$$

$$F_m(T) := \frac{3.81 \cdot 10^3 \cdot D_{\max}^3 \cdot \sqrt{\frac{T}{M_{\text{mix}}}} \cdot \text{cm} \cdot \text{gm}^{0.5}}{a \cdot P_a \cdot \text{K}^{0.5} \cdot \text{mole}^{0.5} \cdot \text{sec}}$$

$$L_{\text{mix}}(T) := (F_c + F_m(T)) \cdot (P_{\text{mix}} - P_d) \cdot \frac{P_a}{P_{\text{mix}}}$$

$$T_F(T) := \left[ (T \cdot F - 273 \cdot K) \cdot \frac{9}{5 \cdot K} + 32 \right]$$

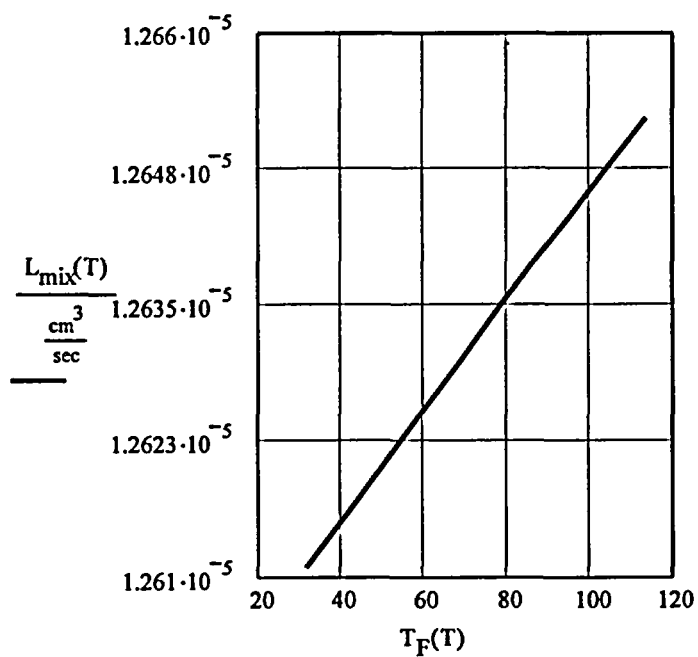


Fig. 4.5 - Allowable R-134a/Air Mixture Test Leakage,  $\text{cm}^3/\text{sec}$ , versus test temperature, deg.F

The R-134a component of this leak rate can be determined by multiplying the leak rate of the mixture by the ratio of the R-134a partial pressure to the total pressure of the mix, as follows.

$$R_{134a}(T) := L_{mix}(T) \cdot \frac{P_{R134a}}{P_{mix}}$$

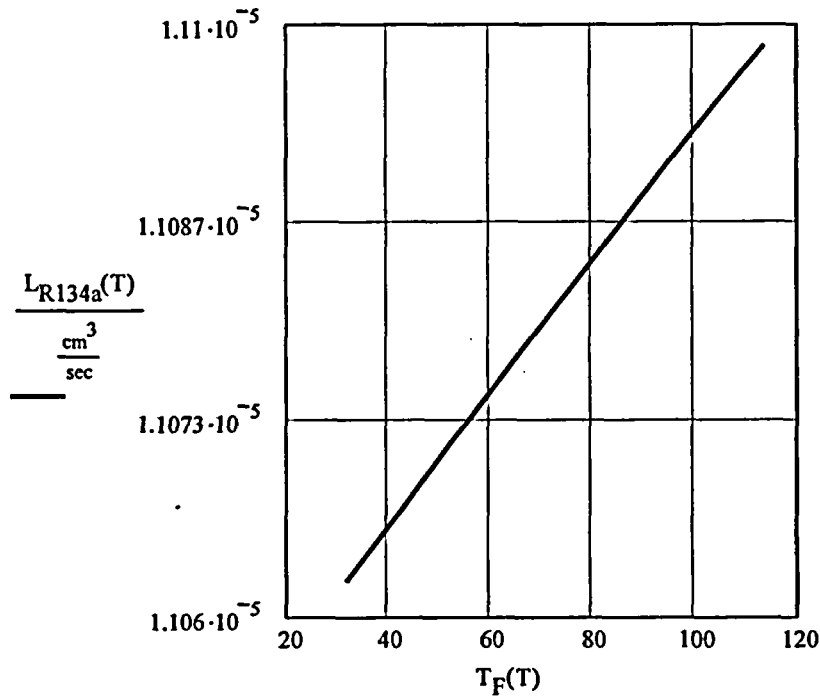


Fig. 4.6 - Allowable R-134a test leakage, cm³/sec versus test temperature, deg.F

Determine the equivalent mass flow rate for  $L_{R134a}$  in oz/yr:

Ideal Gas Law

$$N(T) := \frac{P_{R134a} V}{R_o \cdot T}$$

where,

$$R_o := \frac{82.05 \text{ cm}^3 \cdot \text{atm}}{\text{mole} \cdot \text{K}}$$

This data can then be used to convert the volumetric leak rate for R-134a calculated above to a mass leak rate. By dividing  $N$  by  $V$ , the number of moles per unit volume can be multiplied by the molecular weight of the gas and the maximum allowable volumetric leak rate to determine the maximum allowable mass leak rate, as a function of test temperature as shown in the graph

below. The conversion from grams per second to ounces per year is also shown below.

$$L(T) := L_{R134a}(T) \cdot \frac{N(T)}{V} \cdot M_{R134a} \cdot \frac{yr}{oz}$$

$$\frac{gm}{sec} = 1.113 \times 10^6 \frac{oz}{yr} \quad \text{Conversion of gm/sec to oz/yr}$$

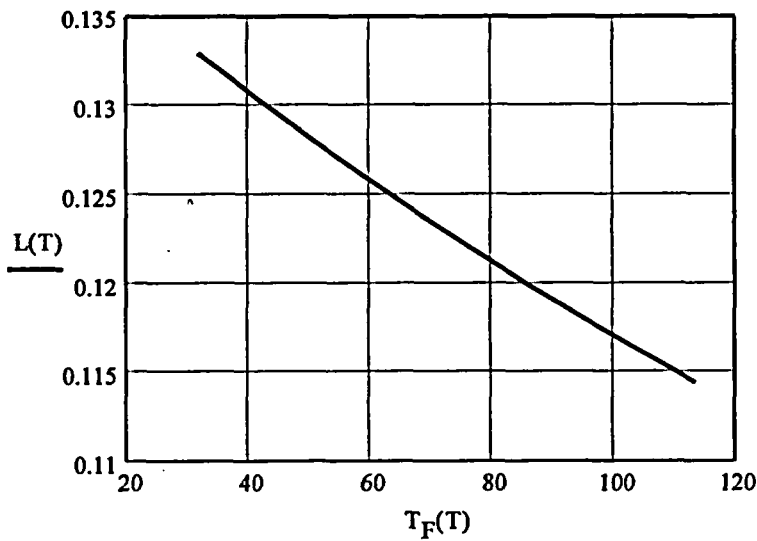


Fig. 4.7 - Allowable R-134a test leakage, oz/yr, versus test temperature, deg.F

The graph above can be used to determine the allowable leak rate based on the temperature at the time of the test. According to ANSI N14.5 methodology, the maximum allowable leak rate must be divided by 2 to determine the minimum sensitivity for the test. A graph of the required sensitivity in oz/yr is presented below:

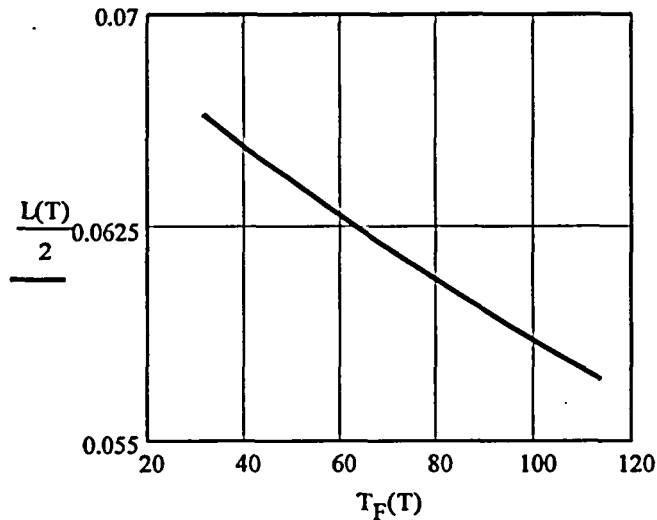


Fig. 4.8 - Allowable R-134a test leakage sensitivity, oz/yr, versus test temperature, deg.F

The values presented in Fig. 4.8 should be used to determine the sensitivity to calibrate the leak detector prior to the test.

#### 4.7 References

1. Weast, Robert C. and Astle, Melvin J., Handbook of Chemistry and Physics, 63<sup>rd</sup> Edition, CRC Press, 1982.
2. Van Wylen, Gordon J. and Sonntag, Richard E., Fundamentals of Classical Thermodynamics. Second Edition, John Wiley and Sons, Inc., 1973.
3. Thomas, Lindon C., Heat Transfer – Professional Version, Prentice-Hall, Inc., 1993.
4. American National Standard for Leakage Tests on Packages for Shipment of Radioactive Materials, American National Standards Institute, Inc., New York, ANSI N14.5-1997, 1997.

# **Appendix 4.1**

## **Properties of R-134a**





Suva:  
refrigerants

# DuPont HFC-134a

Properties, Uses,  
Storage and Handling

- Suva 134a refrigerant
- Suva 134a (Auto) refrigerant
- Formacel<sup>®</sup> Z-4 foam expansion agent
- Dymel<sup>®</sup> 134a aerosol propellant
- Dymel 134a/P aerosol propellant

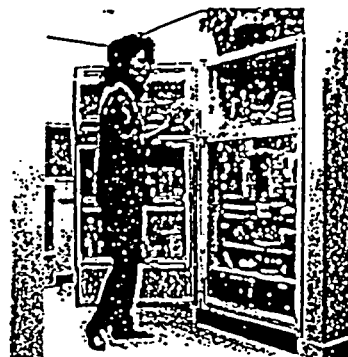


Table 2  
Physical Properties of HFC-134a

Physical Properties	Units	HFC-134a
Chemical Name	—	Ethane, 1,1,1,2-Tetrafluoro..
Chemical Formula	—	$CH_2FCF_3$
Molecular Weight	—	102.03
Boiling Point at 1 atm (101.3 kPa or 1.013 bar)	°C °F	-26.1 -14.9
Freezing Point	°C °F	-103.3 -153.9
Critical Temperature	°C °F	101.1 213.9
Critical Pressure	kPa lb/in. <sup>2</sup> abs	4060 588.9
Critical Volume	m <sup>3</sup> /kg ft <sup>3</sup> /lb	$1.94 \times 10^{-3}$ 0.031
Critical Density	kg/m <sup>3</sup> lb/ft <sup>3</sup>	515.3 32.17
Density (Liquid) at 25°C (77°F)	kg/m <sup>3</sup> lb/ft <sup>3</sup>	1206 75.28
Density (Saturated Vapor) at Boiling Point	kg/m <sup>3</sup> lb/ft <sup>3</sup>	5.25 0.328
Heat Capacity (Liquid) at 25°C (77°F)	kJ/kg·K or Btu/(lb) (°F)	1.44 0.339
Heat Capacity (Vapor) at Constant Pressure at 25°C (77°F) and 1 atm (101.3 kPa or 1.013 bar)	kJ/kg·K or Btu/(lb) (°F)	0.852 0.204
Vapor Pressure at 25°C (77°F)	kPa bar psia	666.1 6.661 96.61
Heat of Vaporization at Boiling Point	kJ/kg Btu/lb	217.2 93.4
Thermal Conductivity at 25°C (77°F) Liquid	W/m·K Btu/hr·ft·°F	0.0824 0.0478
Vapor at 1 atm (101.3 kPa or 1.013 bar)	W/m·K Btu/hr·ft·°F	0.0145 0.00836
Viscosity at 25°C (77°F) Liquid	mPa·S (cP)	0.202
Vapor at 1 atm (101.3 kPa or 1.013 bar)	mPa·S (cP)	0.012
Solubility of HFC-134a in Water at 25°C (77°F) and 1 atm (101.3 kPa or 1.013 bar)	wt %	0.15
Solubility of Water in HFC-134a at 25°C (77°F)	wt %	0.11
Flammability Limits in Air at 1 atm (101.3 kPa or 1.013 bar)	vol %	None
Autoignition Temperature	°C °F	770 1418
Ozone Depletion Potential	—	0
Halocarbon Global Warming Potential (HGWP) (For CFC-11, HGWP = 1)	—	0.28
Global Warming Potential (GWP) (100 yr. ITH. For CO <sub>2</sub> , GWP = 1)	—	1200
TSCA Inventory Status	—	Reported/Included
Toxicity AEL <sup>(a)</sup> (8- and 12-hr TWA)	ppm (v/v)	1000

<sup>(a)</sup>AEL (Acceptable Exposure Limit) is an airborne inhalation exposure limit established by DuPont that specifies time-weighted average concentrations to which nearly all workers may be repeatedly exposed without adverse effects.

Note: kPa is absolute pressure.



# SUVA

Revision 6  
April 2003

## REFRIGERANTS

ART - 1

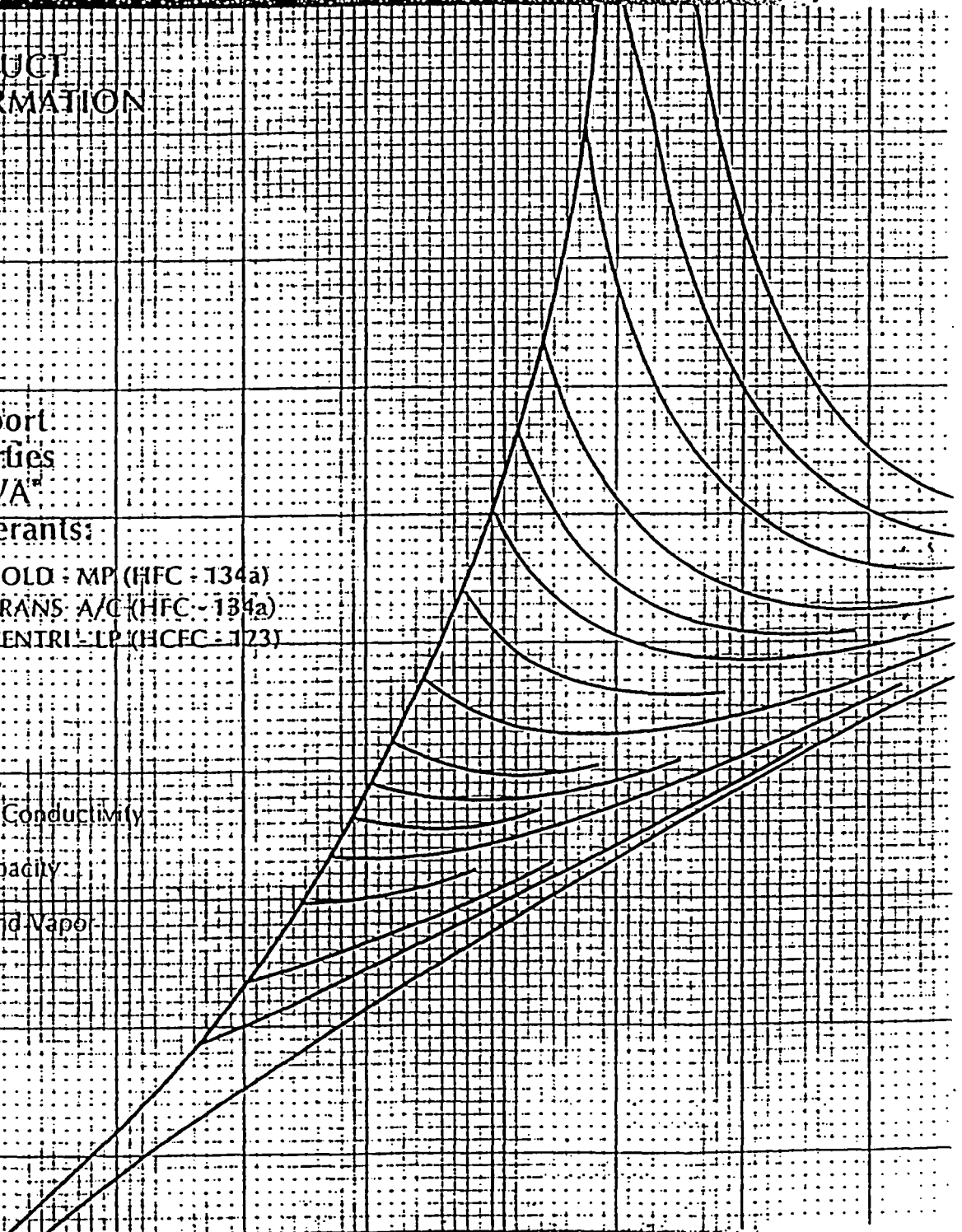
### PRODUCT INFORMATION

### Transport Properties of SUVA<sup>®</sup>

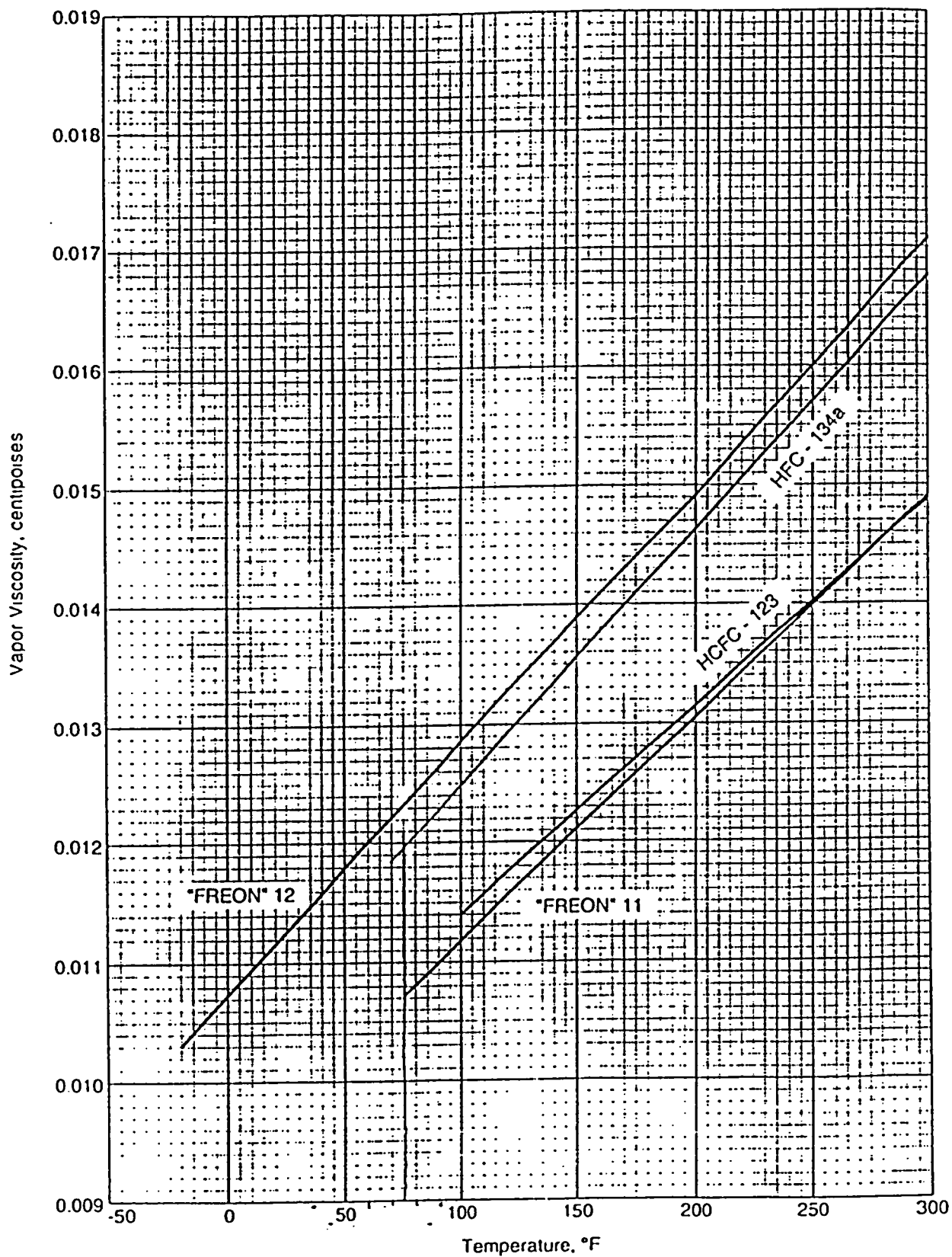
#### Refrigerants:

- SUVA<sup>®</sup> COLD - MP (HFC - 134a)
- SUVA<sup>®</sup> TRANS - A/C (HFC - 134a)
- SUVA<sup>®</sup> CENTRI - LP (HCEC - 123)

### Viscosity, Thermal Conductivity and Heat Capacity for the Liquid and Vapor



### Vapor Viscosity at Atmospheric Pressure



## 8.0 ACCEPTANCE TESTS AND MAINTENANCE

### 8.1 Acceptance Tests

Prior to the first use of the CNS 8-120B package, the following tests and evaluations will be performed.

#### 8.1.1 Visual Examination

The package will be examined visually for any adverse conditions in materials or fabrication. Welds shall be examined for compliance to the drawings. Weld integrity shall be verified by visual examination and magnetic particle or dye penetrant. NDE examinations shall be performed by an ASME Certified inspector. Acceptance criteria for NDE shall be according to ASME Code Section III, Div. 1-Section NB5342 or NB5352 as applicable.

#### 8.1.2 Structural Tests

No structural testing is required.

#### 8.1.3 Leak Tests

This test shall be performed prior to acceptance and operation of a newly fabricated package in accordance with ASTM E-427 using a leak detector capable of detecting the applicable leak rates specified in Figures 4.3 and 4.7 in Chapter 4. Calibration of the leak detector shall be performed using a leak rate standard traceable to NIST. The standard's setting shall correspond to the approved leak rates specified in Figures 4.3 and 4.7 in Chapter 4.

All four containment boundary penetrations must be tested.

- The volume above the vent port Stat-O-Seal
- The volume between the drain line plug and interior of the cask
- The annulus between the o-ring seals of the primary lid
- The annulus between the o-ring seals of the secondary lid

All four of these volumes must be evacuated to a minimum vacuum of 20" Hg, and then be pressurized to a minimum pressure of 25 psig with pure dichlorodifluoromethane (R-12) or 1,1,1,2-tetrafluoroethane (R-134a). Use the detector probe to "sniff" the following areas:

- The vent port penetration on the underside of the primary lid
- Around the head of the cap screw that plugs the drain line

- Interior side of the inner o-ring for the primary lid
- Interior side of the inner o-ring for the secondary lid

Leak detection shall be in accordance with the specifications of ASTM E-427.

Any condition, which results in leakage in excess of the applicable values specified in Figures 4.3 and 4.7 in Chapter 4 shall be corrected.

#### 8.1.4 Component Tests

Gaskets and seals will be procured and examined in accordance with the Duratek Quality Assurance Program.

#### 8.1.5 Test for Shielding Integrity

Shielding integrity of the package will be verified by gamma scan or gamma probe methods to assure the package is free of significant voids in the poured lead shield annulus. All gamma scanning will be performed on a 4-inch square or less grid system. The acceptance criteria will be that voids resulting in shield loss in excess of 10 % of the normal lead thickness in the direction measured shall not be acceptable.

#### 8.1.6 Thermal Acceptance Tests

No thermal acceptance testing will be performed on the CNS 8-120B package. Refer to the Thermal Evaluation, Chapter 3.0 of the report.

### 8.2 Maintenance Program

Duratek is committed to an ongoing preventative maintenance program for all shipping packages. The 8-120B package will be subjected to routine and periodic inspections and tests as outlined in this Chapter and Duratek approved procedures.

#### 8.2.1 Routine Maintenance

Unless noted otherwise, for loaded packages containing material greater than "Type A" quantities, each of the following safety related items and functional features shall be visually examined for defects or replacement. Corrective action for defects shall be as noted.

#### 8.2.1.1 Fasteners

The primary and secondary lid bolts shall be visually inspected for defects whenever it is necessary to remove them. Obtain replacement bolts as specified on Drawing No. C-110-E-0007 (current revision) for any bolts that show cracking or other visual signs of distress.

The cap screws for the vent port, test ports and drain shall be visually inspected for defects whenever it is necessary to remove them. Obtain replacement cap screws as specified on Drawing No. C-110-E-0007 (current revision) for any cap screws that show cracking or other visual signs of distress.

#### 8.2.1.2 Gaskets and Seals

##### (A) Primary Lid Seals

The primary lid o-ring seals shall be visually inspected (at any time it is necessary to remove the primary lid) for serviceability ensuring that they are in the proper position and free of cracks, tears, cuts, or discontinuities which may prevent them from sealing properly. The seal seating surfaces shall be visually inspected to ensure that they are free of damage, dirt, gravel, or any foreign matter which might damage the seals. If any significant defects are detected, the seals shall be replaced and/or the seal seating surfaces shall be reworked as necessary to ensure that the lid will seal properly.

##### (B) Secondary Lid Seals

The secondary lid o-ring seals and seating surfaces shall be inspected as specified in Section 8.2.1.2(a) at any time it is necessary to remove the secondary lid. Seal replacement and/or seating surface repair shall be as specified in Section 8.2.1.2(a) if any defects are detected.

##### (C) Vent and Drain Seals

The above seals and seating surfaces shall be inspected as specified in Section 8.2.1.2(a) at any time it is necessary to remove them. Seal replacement and/or seating surface repair shall be as specified in Section 8.2.1.2(a) if any defects are detected.

**8.2.1.3 Painted Surfaces, Identification Markings, and Match Marks Used for Closure Orientation.**

The above items shall be visually inspected to ensure that painted surfaces are in good condition, identification markings are legible, and that match marks used for closure orientation remain legible and are easy to identify.

**8.2.2 Periodic Maintenance**

The following inspections and/or tests shall be performed as specified.

**8.2.2.1 Periodic Leak Tests**

The package will be leak tested as described in Section 8.1.3 after its third use. In addition, the containment system, before actual use for shipment, shall have been leak tested according to Section 8.1.3 within the preceding 12-month period.

Also, before actual use for shipment, all seals shall have been replaced within the preceding 12-month period.

**8.2.2.2 Assembly Verification Leak Test**

This test is required for any shipment of material that is greater than a "Type A" quantity of radioactive material. For content exemptions of these tests, refer to the current Certificate of Compliance No. 9168. The test will verify that the containment system has been assembled properly.

The test will be performed by pressurizing the annulus between the o-ring seals of the primary and secondary lids with air.

In addition, prior to shipment, the vent and drain ports shall be tested by pressurizing the volume above the respective plug and stat-o-seal, anytime they have been removed during the cask loading operation.

The test shall be performed using a pressure gauge, readable without estimation and calibrated to a maximum error of  $\pm 1$  % of full scale.



The test pressure shall normally be 18 psig (up to 20 psig) and the test shall last for 1 hour. The allowable pressure drop shall be 1 psi. Any condition which results in a pressure drop of more than 1 psi will be corrected.

Sensitivity at the test conditions is equivalent to the prescribed procedure sensitivity of  $5 \times 10^{-4} \frac{\text{atm} \cdot \text{cm}^3}{\text{sec}}$  based on dry air at standard conditions as defined in ANSI N14.5-1987 (see Section 4.4 for the determination of the test conditions).

#### 8.2.3 Subsystem Maintenance

The CNS 8-120B package contains no subsystem assemblies.

#### 8.2.4 Valves, Rupture Discs, and Gaskets on Containment Vessel

As a minimum, all seals will be replaced prior to the annual leak test specified in 8.2.2.1.

#### 8.2.5 Shielding

No shielding tests will be performed after acceptance testing unless damage has required repairs affecting shield integrity. Any shield testing which might be required would be in accordance with the original criteria specified in Section 8.1.5.