FRAMATOME ANP

High-Pressure Separate Effect Test Facility- More than 25 Years in Operation

BENSON Facility

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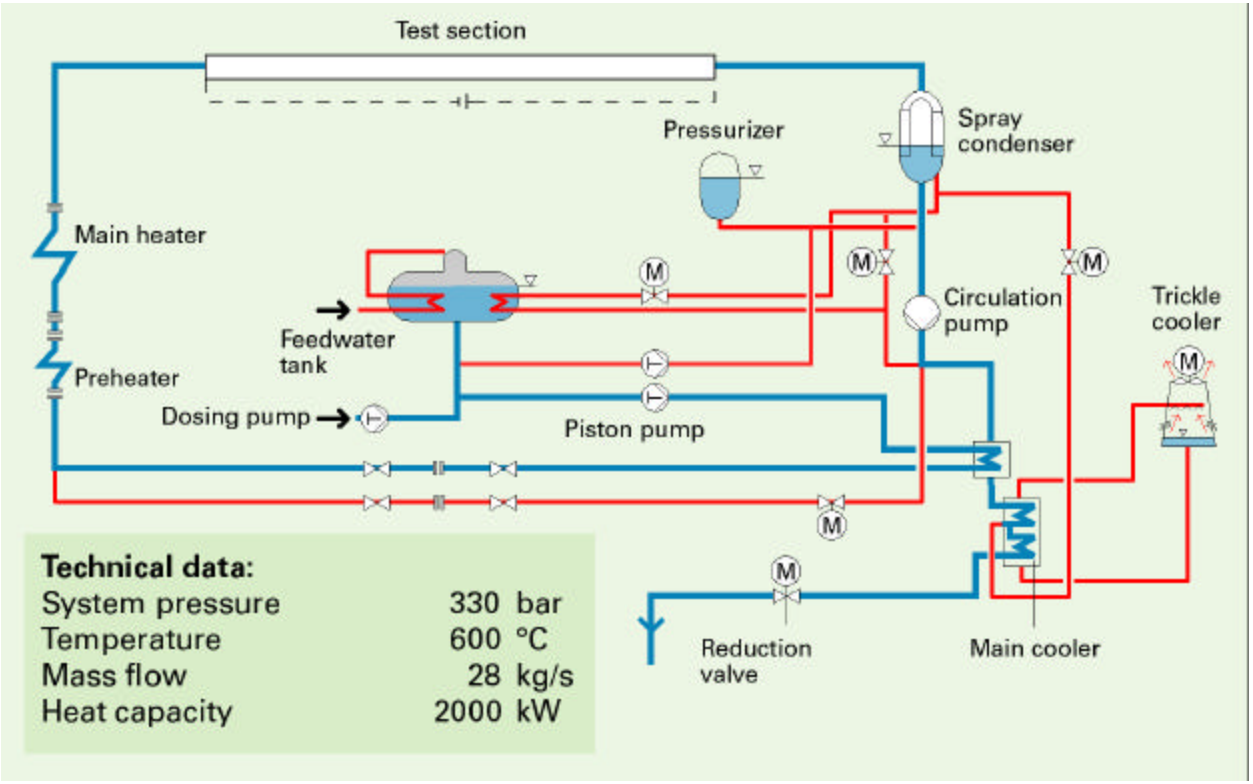
Flow diagram



Technical data:

System pressure	330	bar
Temperature	600	°C
Mass flow	28	kg/s
Heat capacity	2000	kW

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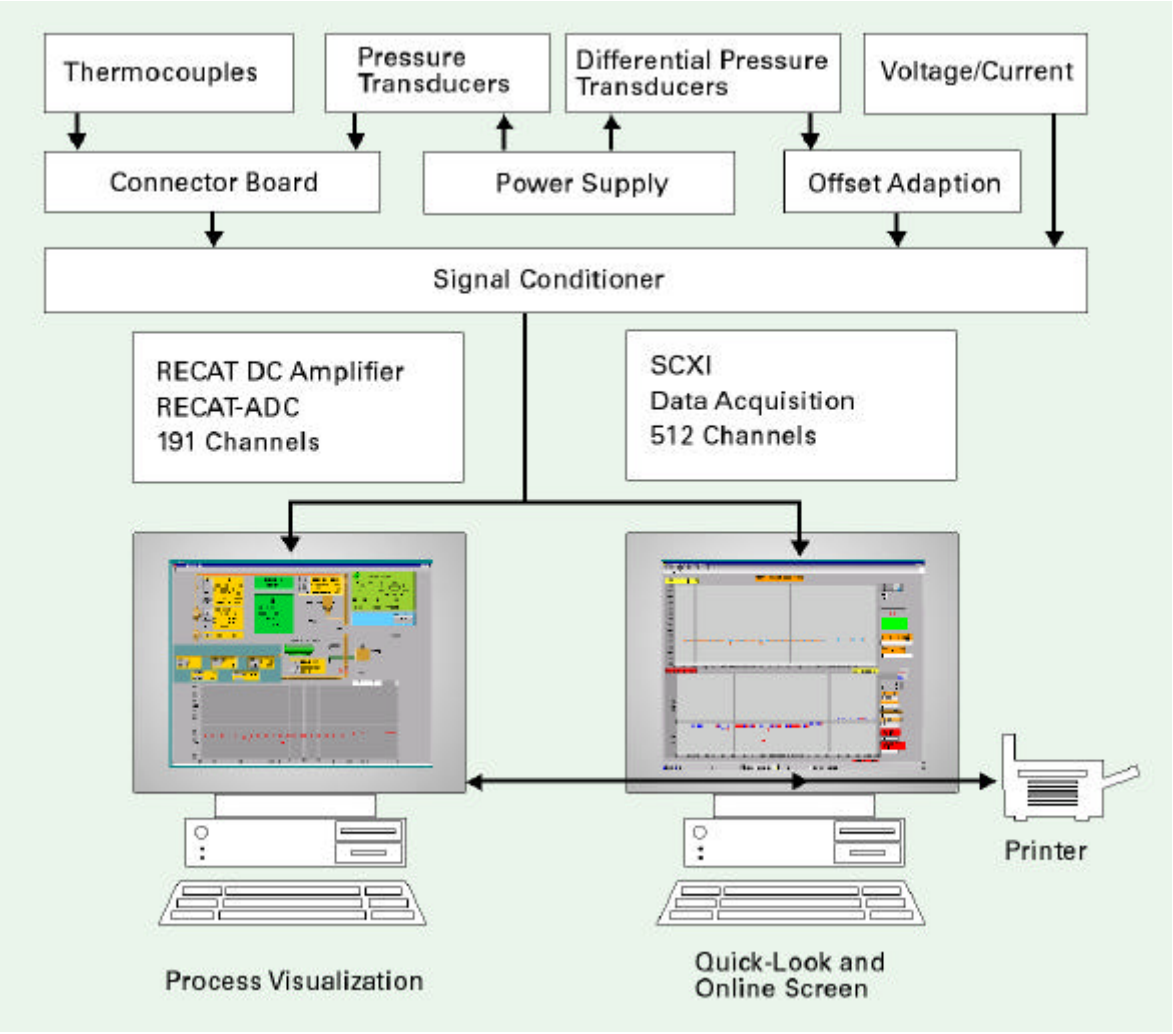
Data acquisition and processing



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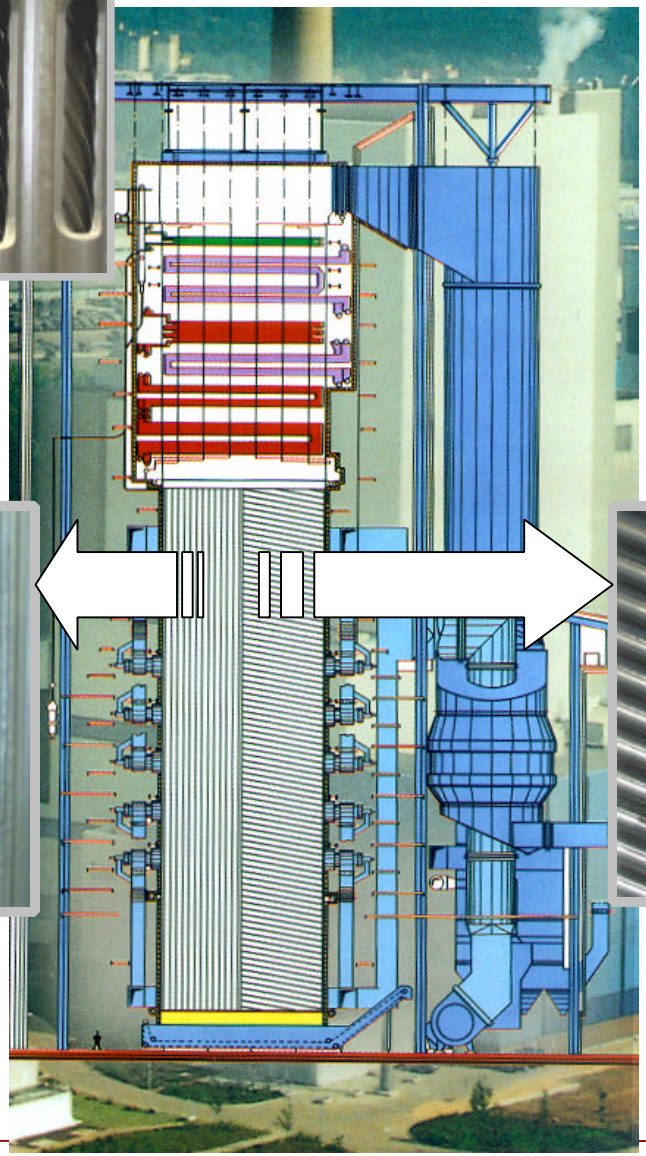
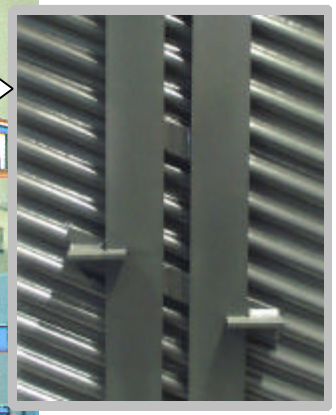
More than 25 years of experiences

Topic of Investigation	Special Test Features
Fossil-Fired Power Generation	
• Heat transfer and pressure loss in tubes for BENSON and drum boilers	Pressures from 25 to 280 bar, steady-state and transient conditions
• Thermal stresses in membrane waterwalls of BENSON boilers	Transient conditions
• Cooldown of hot surfaces (Leidenfrost effect)	Pressures up to critical pressure (220 bar)
• Wet steam measuring system for improved feedwater control at BENSON boilers	Combination of venturi tube and gamma densitometer, pressures up to 220 bar
Nuclear Power Generation	
• GAP cooling between debris crust and reactor pressure vessel wall (TMI 2 accident)	Complex test setup; pressures up to 115 bar; gaps of 1 to 10 mm
• Heat transfer performance of a safety condenser for new pressurized water reactors	Complex test setup with elevations scaled 1:1; 600 kW/m^2, $\leq$ 40 kg/m^{2}s, $\leq$1.35 bar on secondary side
• Cladding tube temperatures in fuel assembly with various spacer grid designs	Testing of production-type components under realistic PWR conditions
• Influence of fouling on secondary-side heat transfer in a steam generator tube that has been in service	Testing under realistic conditions; comparisons with archived tube and chemically cleaned tube formerly in service
• Leakage rates from real cracks	Test object with real cracks (bending stress, fatigue); pressures up to 160 bar; 0 to 70 K subcooling
Power Plant concepts for Direct Solar Steam Generation	
• Heat transfer in absorber tubes - Forced-flow once-through concept - Injection concept - Recirculation concept	30-m-long test tube tube diameters of 50, 65 and 85 mm, pressures up to 100 bar
Thermally Enhanced Heavy Oil Recovery	
• Wet steam piping network	Diameters of 25 and 50 mm, up to 100 bar
• Robust measuring techniques for application in the field	Gamma densitometer, venturi and pitot tubes
Water Chemistry	
• Formation of protective magnetite layers	BENSON boiler conditions
• Behavior of protective layers in the event of thermal shocks	Transient conditions
• Material loss due to erosion corrosion	Conditions of fossil-fired boilers and nuclear plant secondary cycles
Other topics	
• Determination of material properties of components made of nickel alloys	Steam generator tubes made from Inconel 690, Incoloy 800, PWR conditions
• Performance tests for heat exchangers of rail vehicles	Traction motor heat exchanger for electric multiple unit trains

Typical BENSON Steam Generator

Furnace Walls of Vertical Rifled Tubing





Furnace Walls of Spiral-Wound Smooth Tubing

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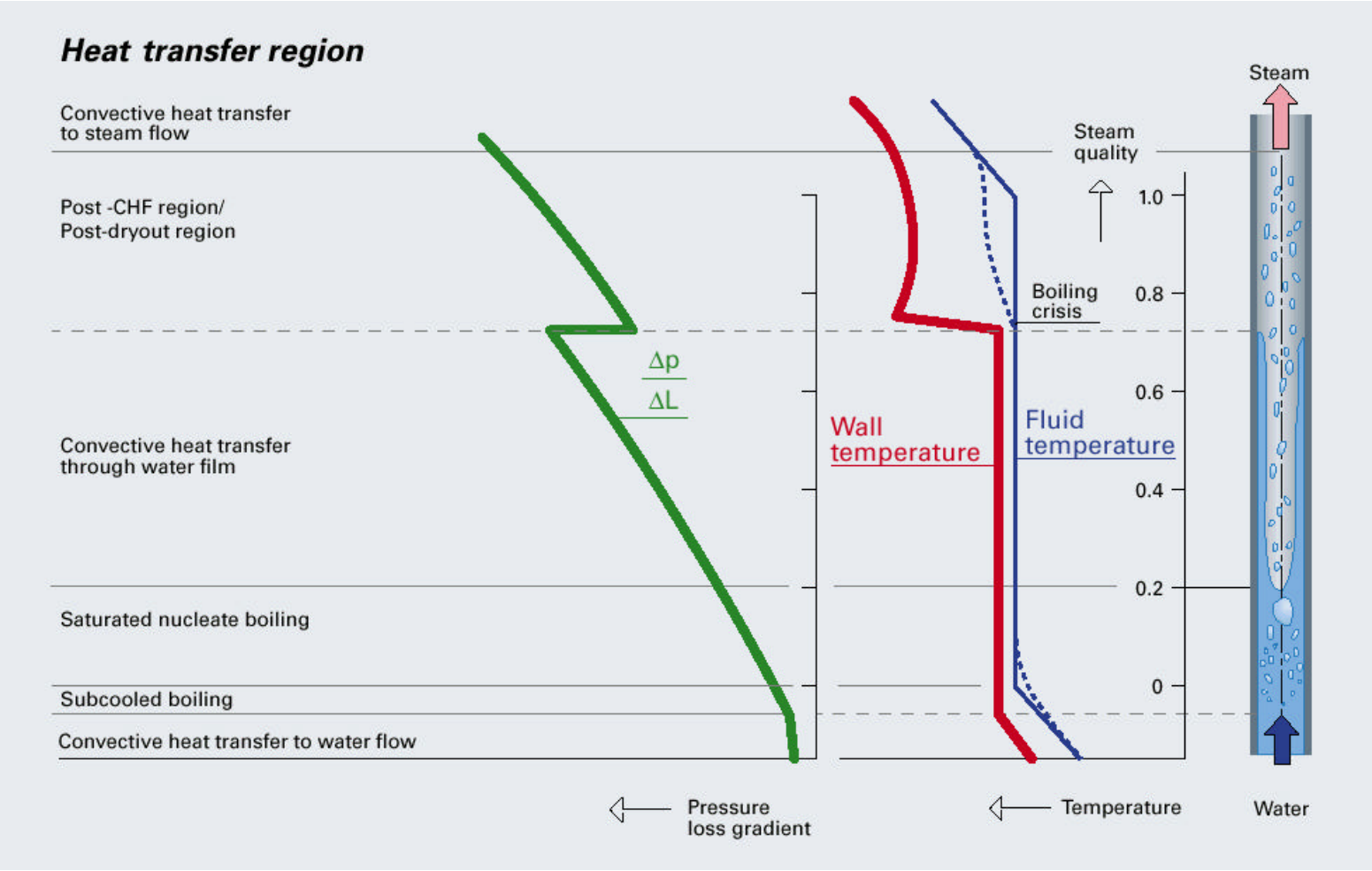
Wall temperature and pressure loss in a uniformly heated, vertical, smooth boiler tube



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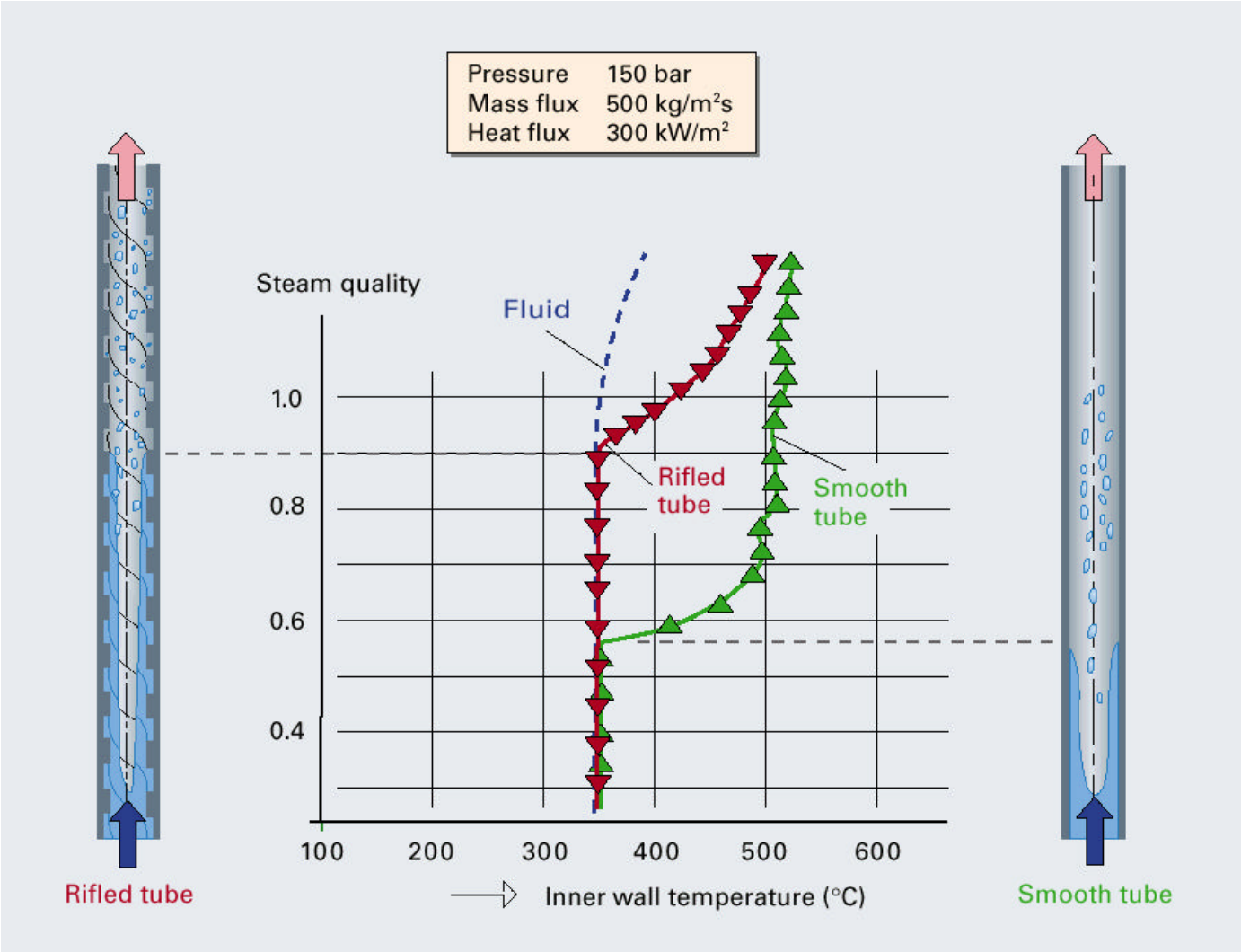
Wall Temperatures in Smooth and Rifled Boiler Tube



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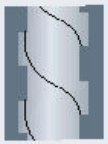
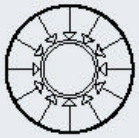
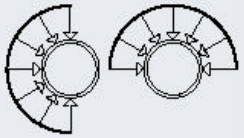
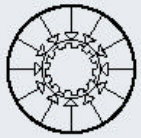
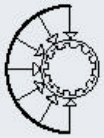
Overview of heat transfer tests for BENSON steam generator tubes

Tube Geometry					
Number of measurements		> 100.000		> 160.000	
Heating		 uniform	 one-side	 uniform	 one-side
Tube orientation	vertical	●	●	●	●
	inclined	●		●	●
	horizontal	●	● ●	●	

Test parameter

Pressure	$25 \leq p \leq 280$ bar
Mass flux	$100 \leq \dot{m} \leq 2500$ kg/m²s
Heat flux	$0 \leq \dot{q} \leq 950$ kW/m²
Tube inner diameter	$8 \leq d \leq 50$ mm

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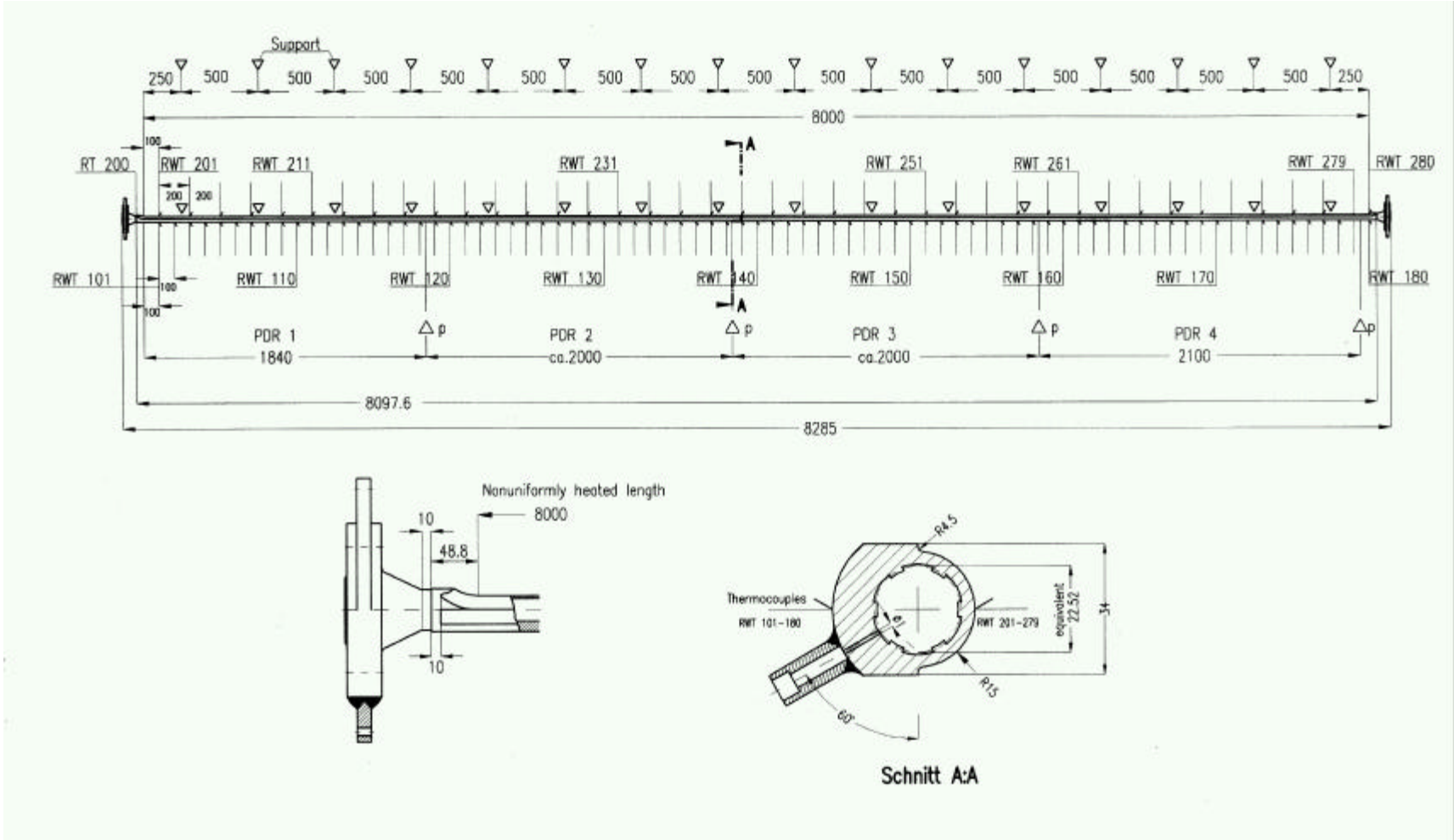
Typical test tube



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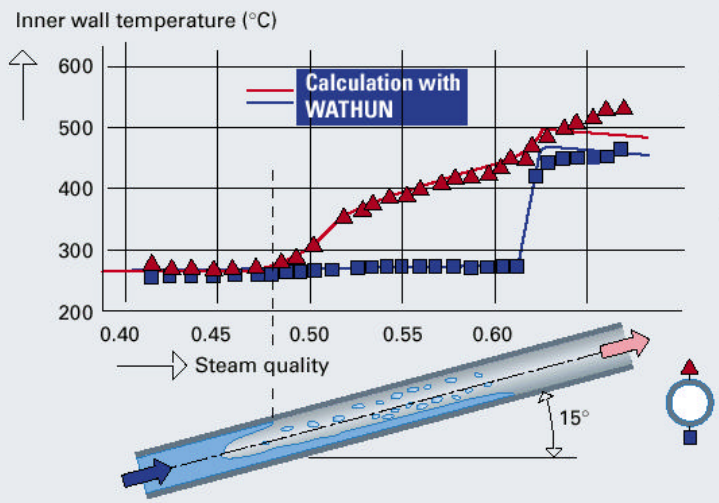
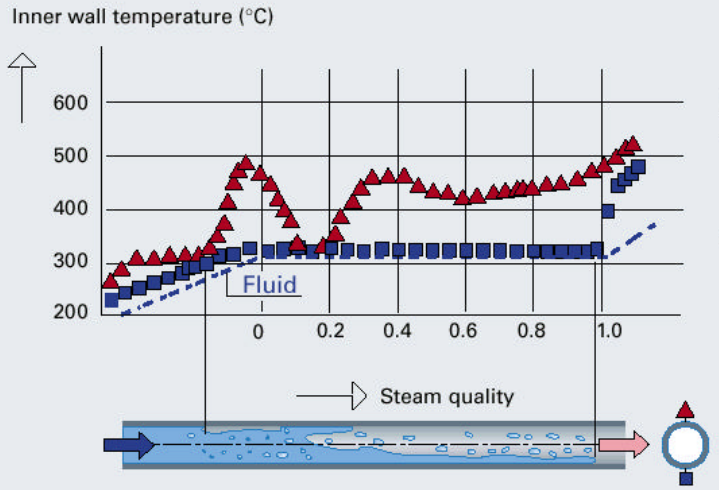
Effect of gravity on heat transfer in inclined and horizontal smooth tubes



Inclined tube

Pressure	50	bar
Mass flux	1000	kg/m²s
Heat flux	400	kW/m²
Tube inner diameter	24.3	mm



Horizontal tube

Pressure	100	bar
Mass flux	500	kg/m²s
Heat flux	300	kW/m²
Tube inner diameter	24.3	mm

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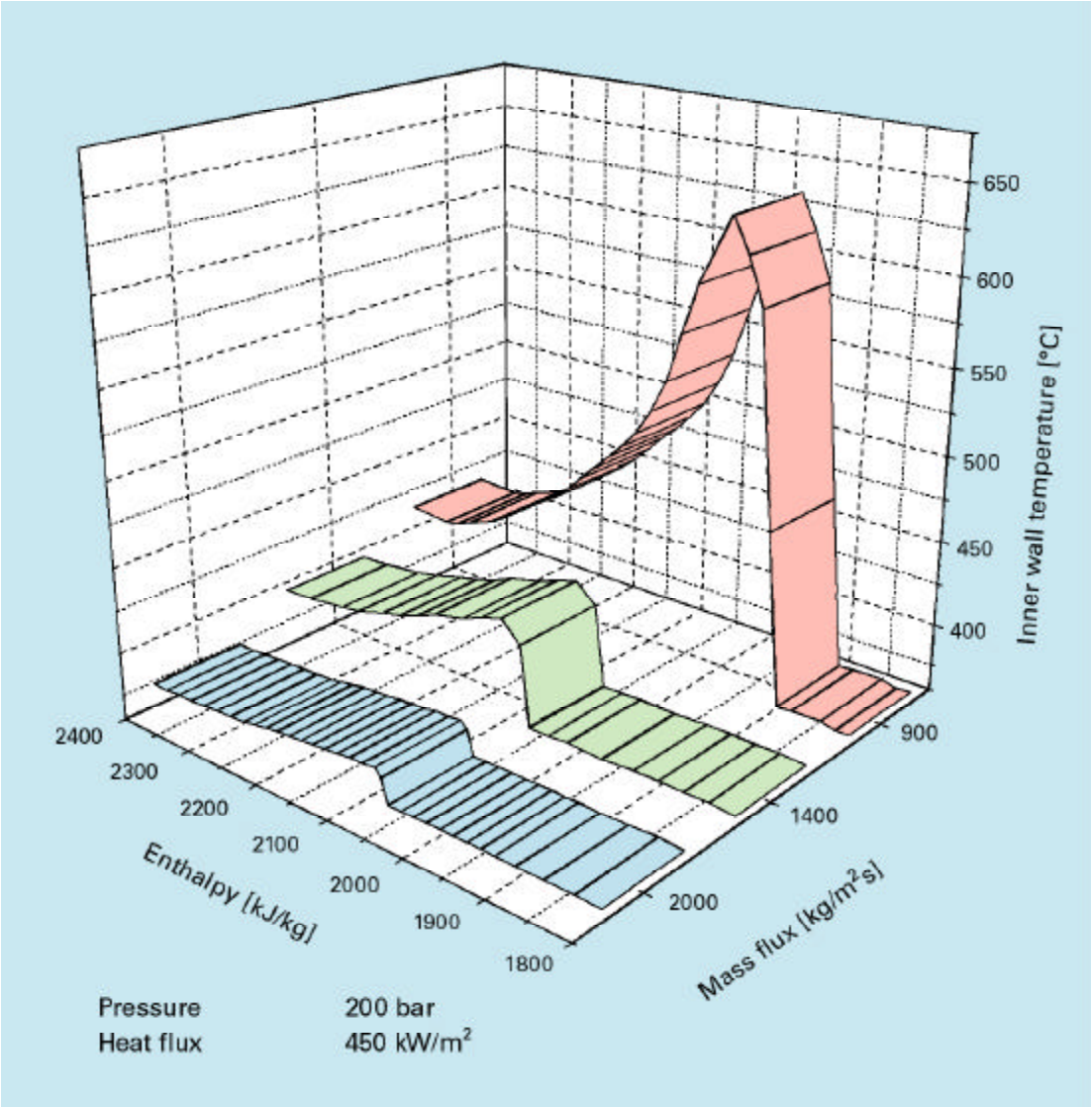
Effect of the pressure



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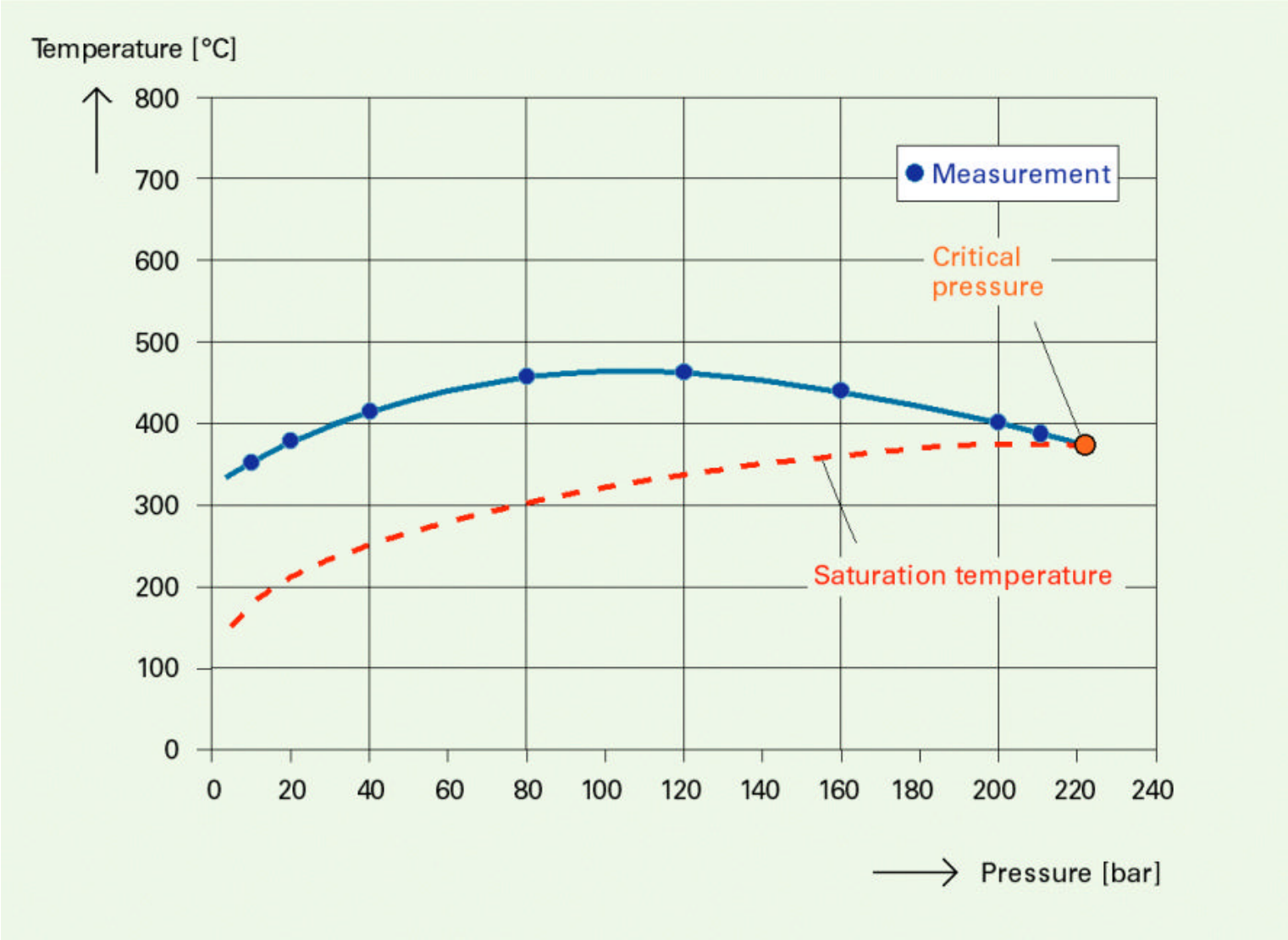
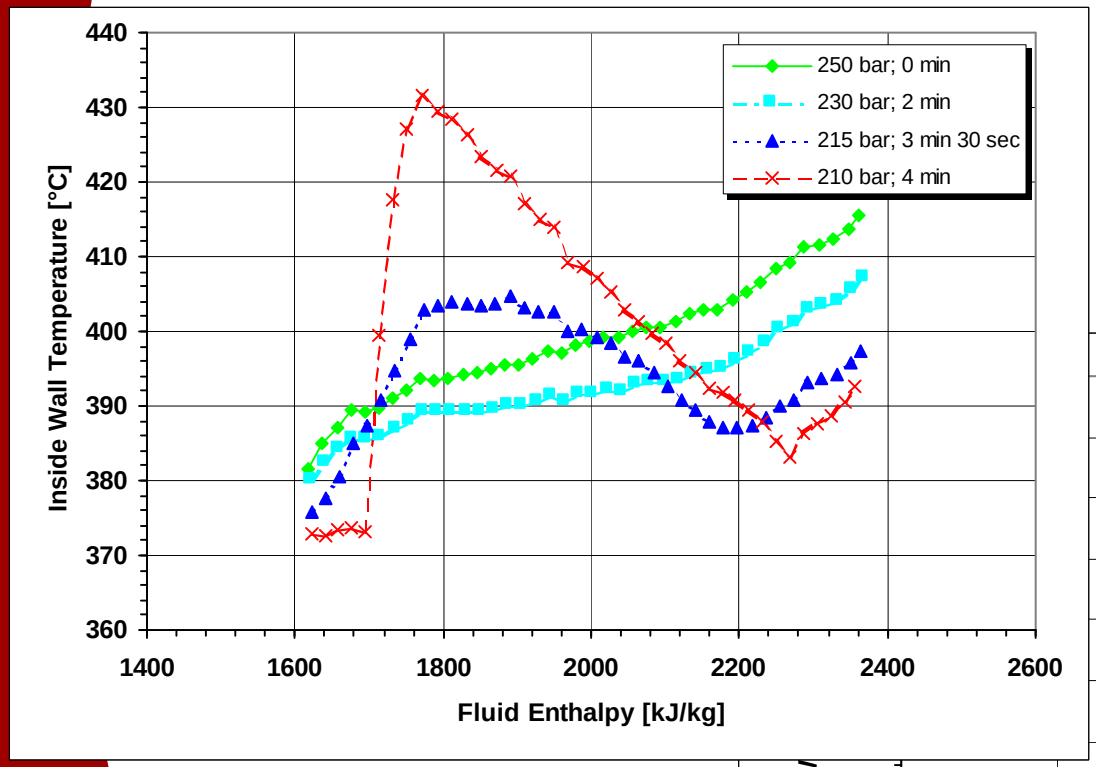
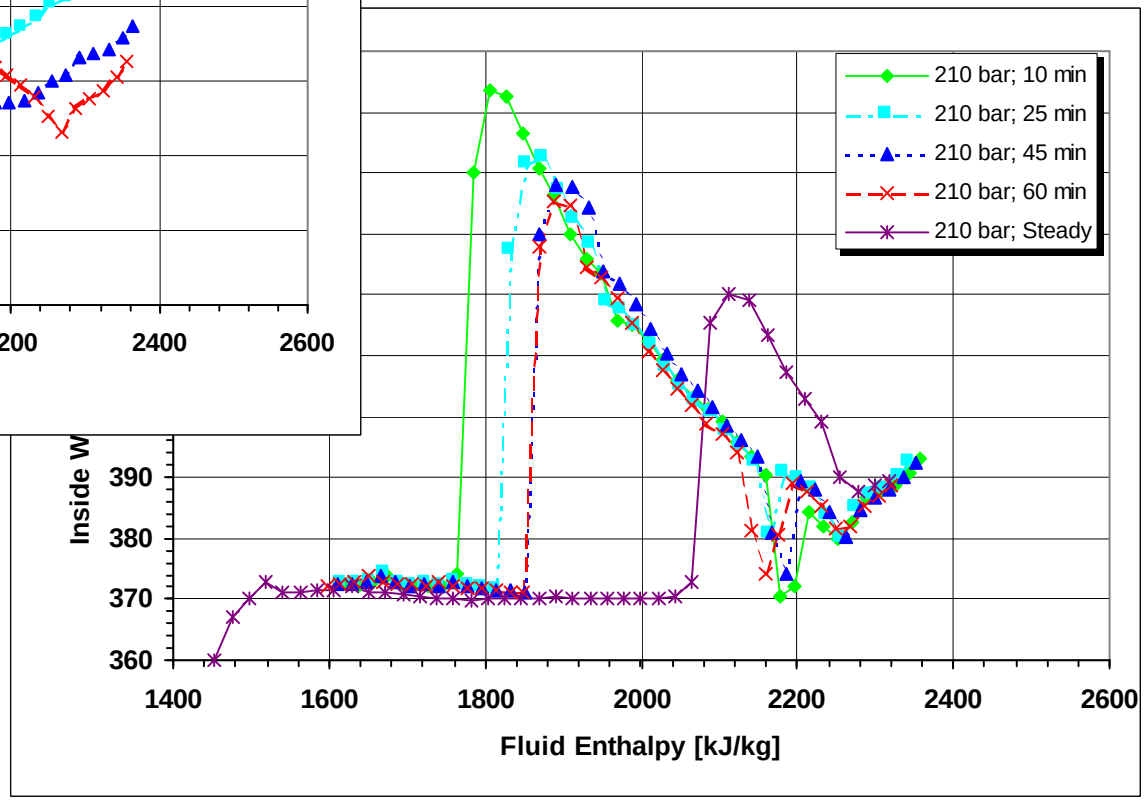
Leidenfrost temperature for forced convection as a function of pressure



Transient tests from super- to sub-critical conditions





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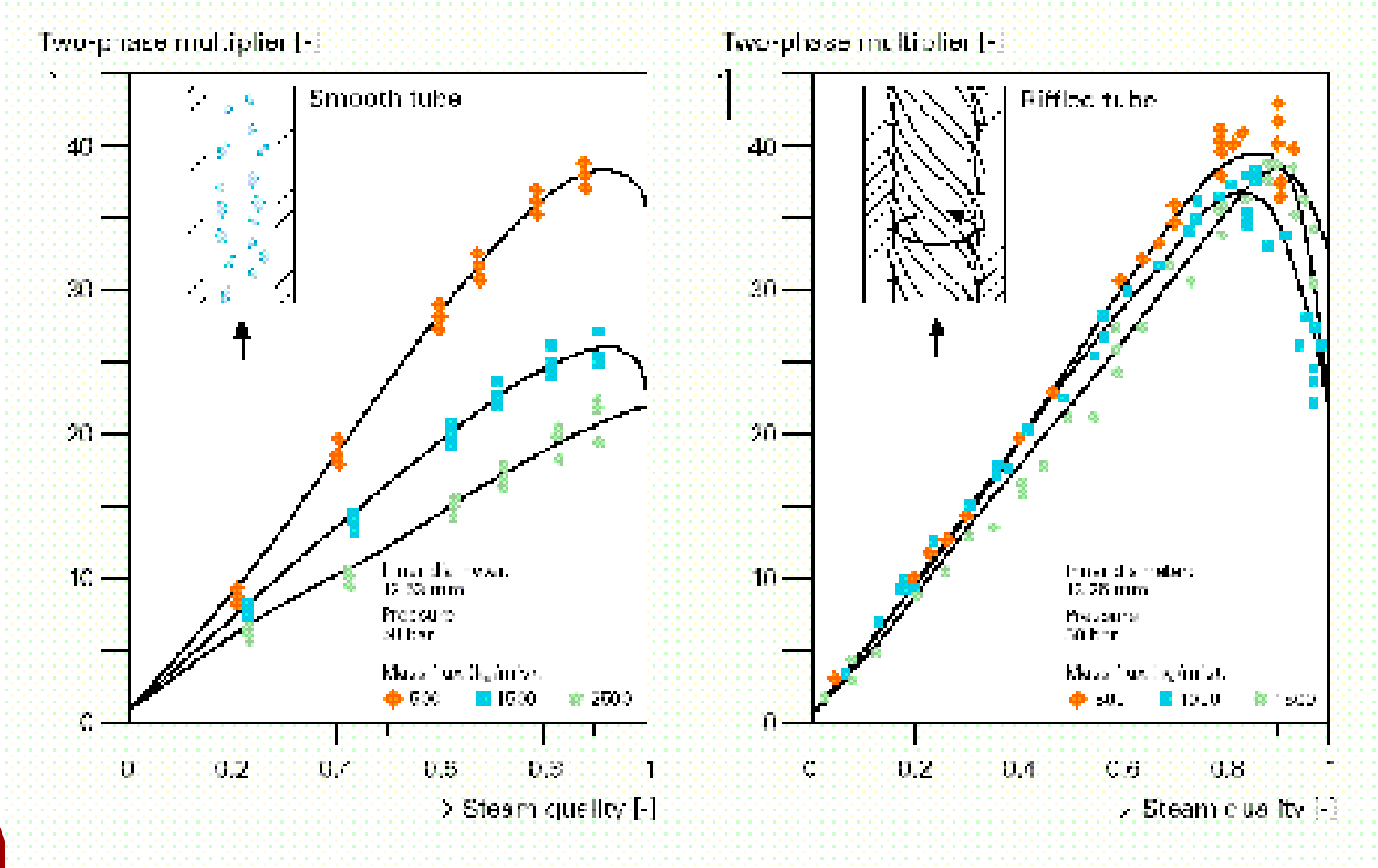
Two-phase flow multiplieres for smooth and rifled tubes



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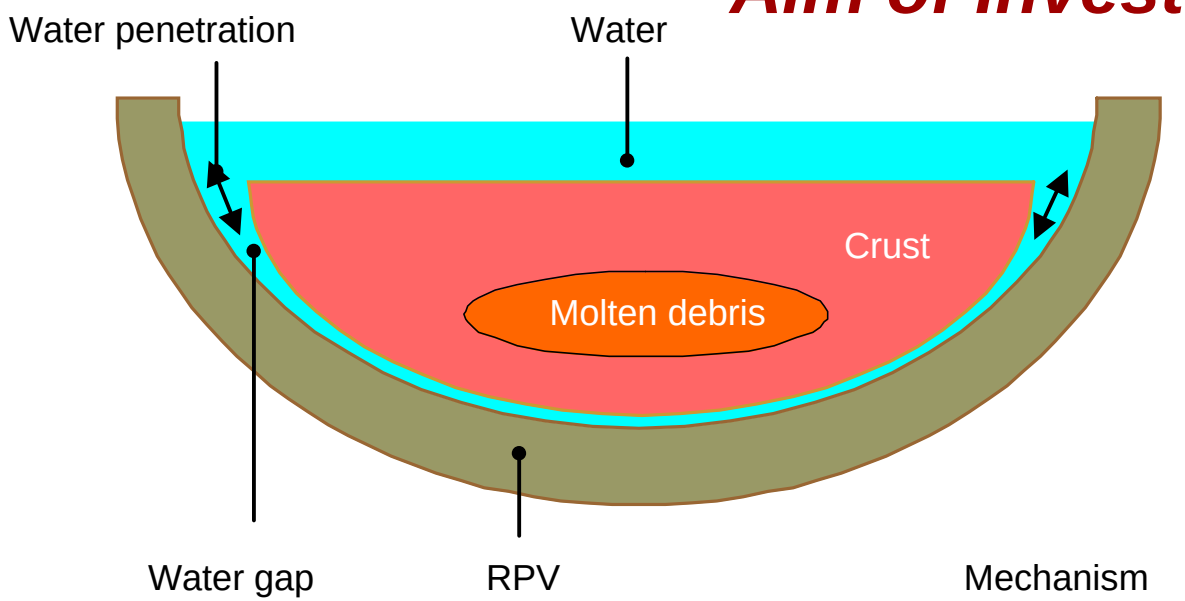
Gap Cooling Process ; TMI-2 Aim of Investigation



Predition of today's codes:

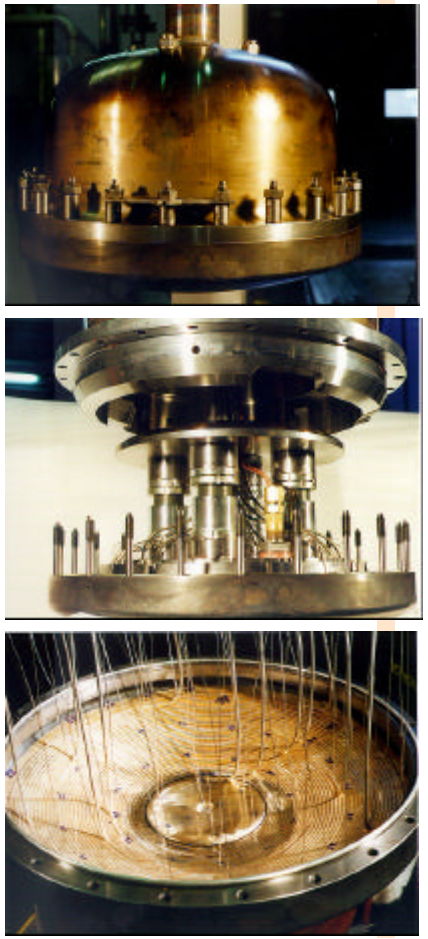
RPV failure due to high material temperatures

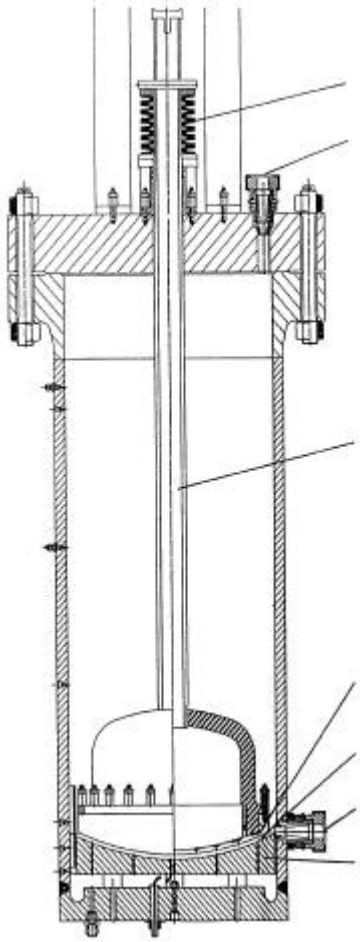
TMI-2 Accident:

No RPV failure due to sufficient cooling of the relocated debris

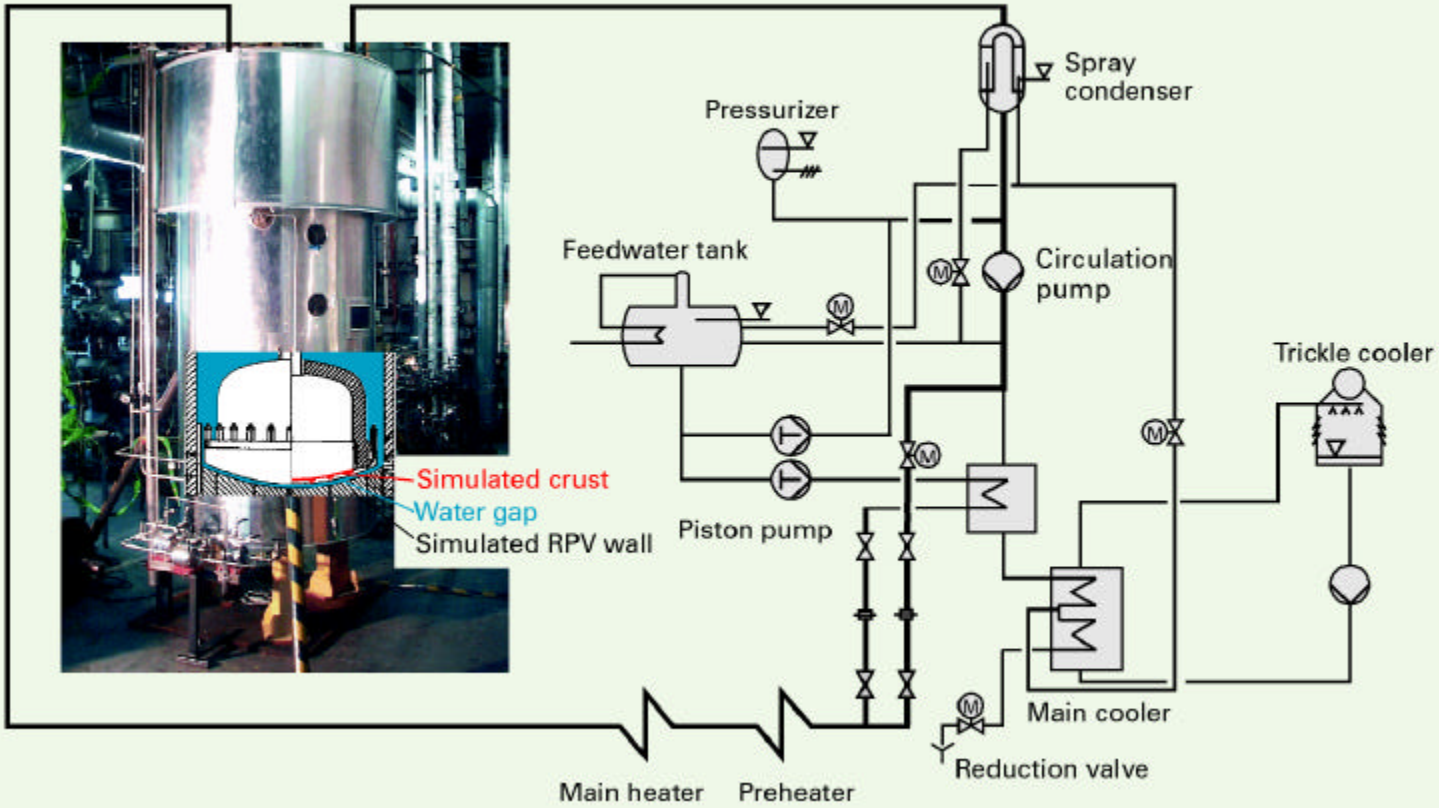
- Simulation of the accident taking into accout a gap cooling mechanism related to the water availability in the lower plenum
- Investigation of conditions which are conductive to efficient steam/water cooling (geysering etc.) crucial for RPV integrity

Gap Cooling Process ; TMI-2 Test Apparatus





Gap Cooling Process ; TMI-2 Test Apparatus intgrated in the BENSON Facility



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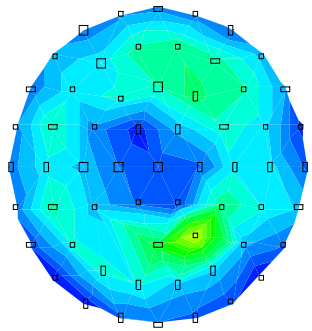
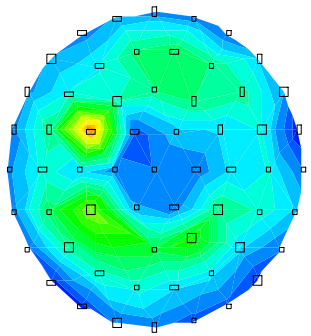
Gap Cooling Process ; TMI-2
Time Sequence During the Transition to Boiling Crisis

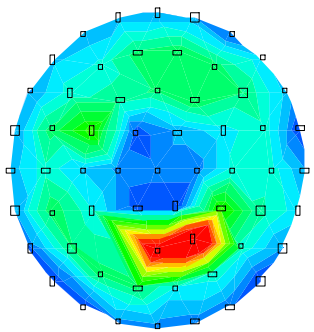
Time: 4:49:57 pm
Heat flux: 434.6 kW/m^2



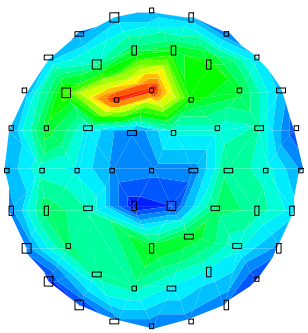
4:50:40 pm
449.9 kW/m^2



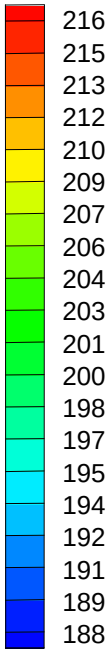
4:51:59 pm
473.7 kW/m^2



4:52:10 pm
474.0 kW/m^2



Temperature
[°C]



Time sequence

Gap width:	5.0	mm
Water submersion:	1.3	m
Pressure:	10	bar
Subcooling:	0.0	K

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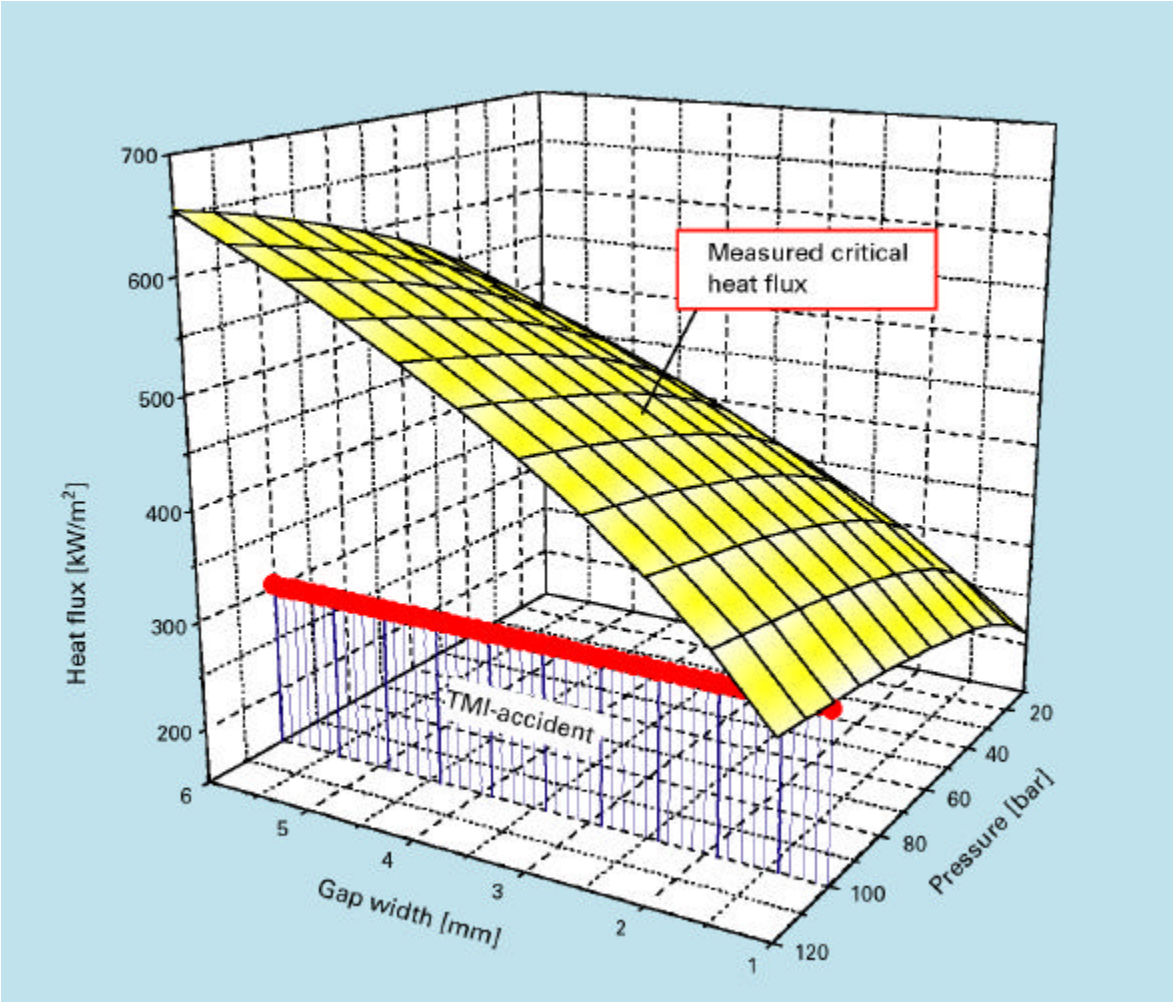
Critical heat flux in a gap versus decay heat TMI-2



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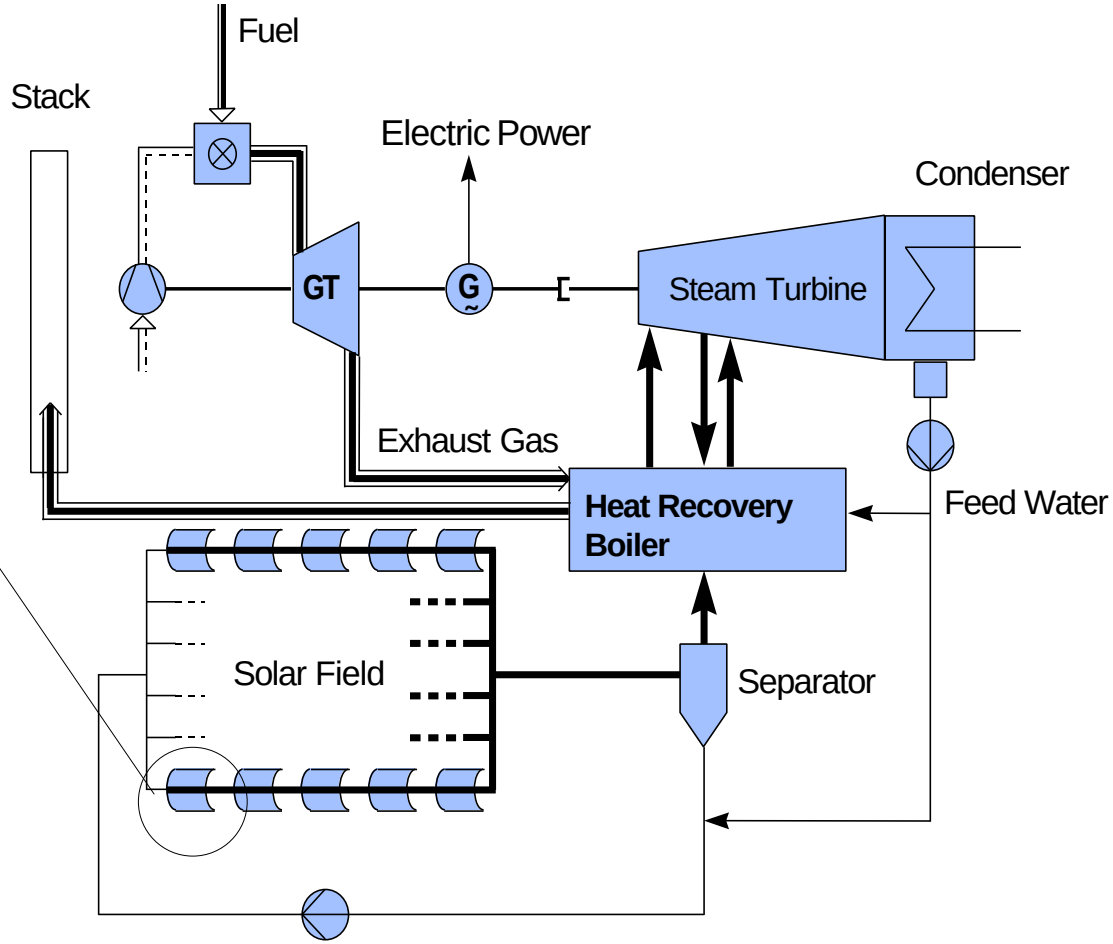
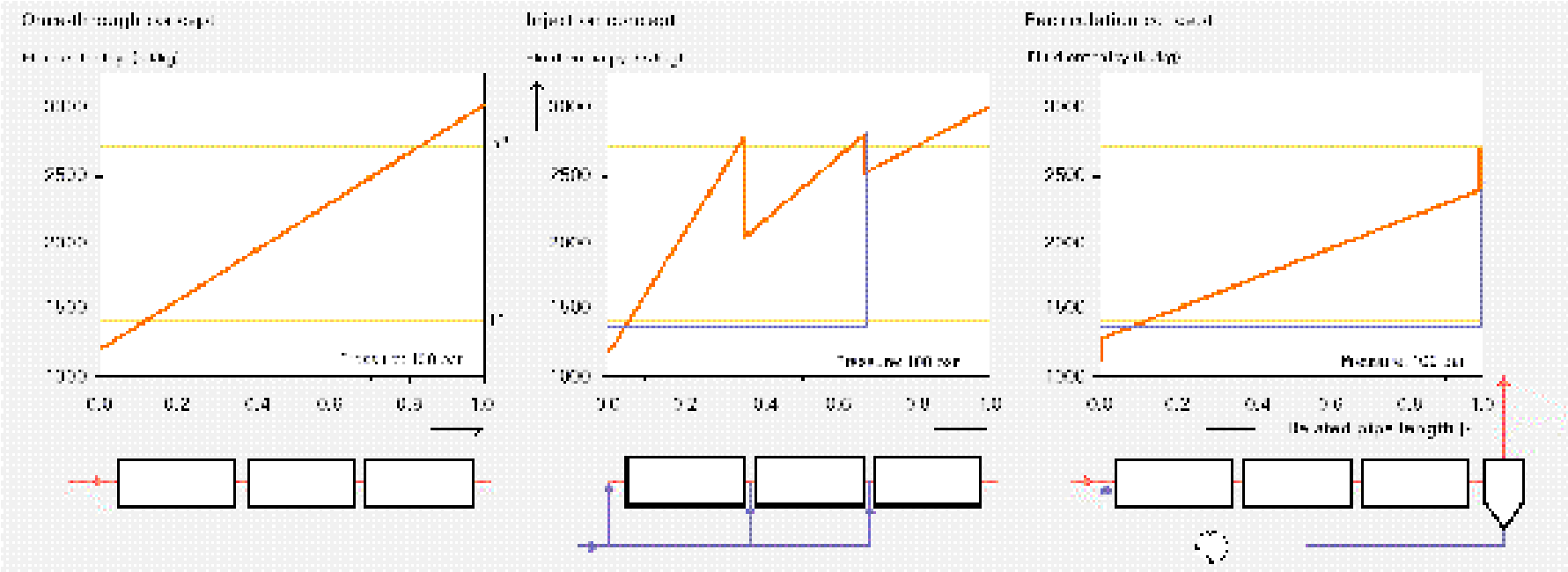
Heat Transfer Behavior in Absorber Tubes of Solar Power Plants with Direct Steam Generation



Solar field concepts



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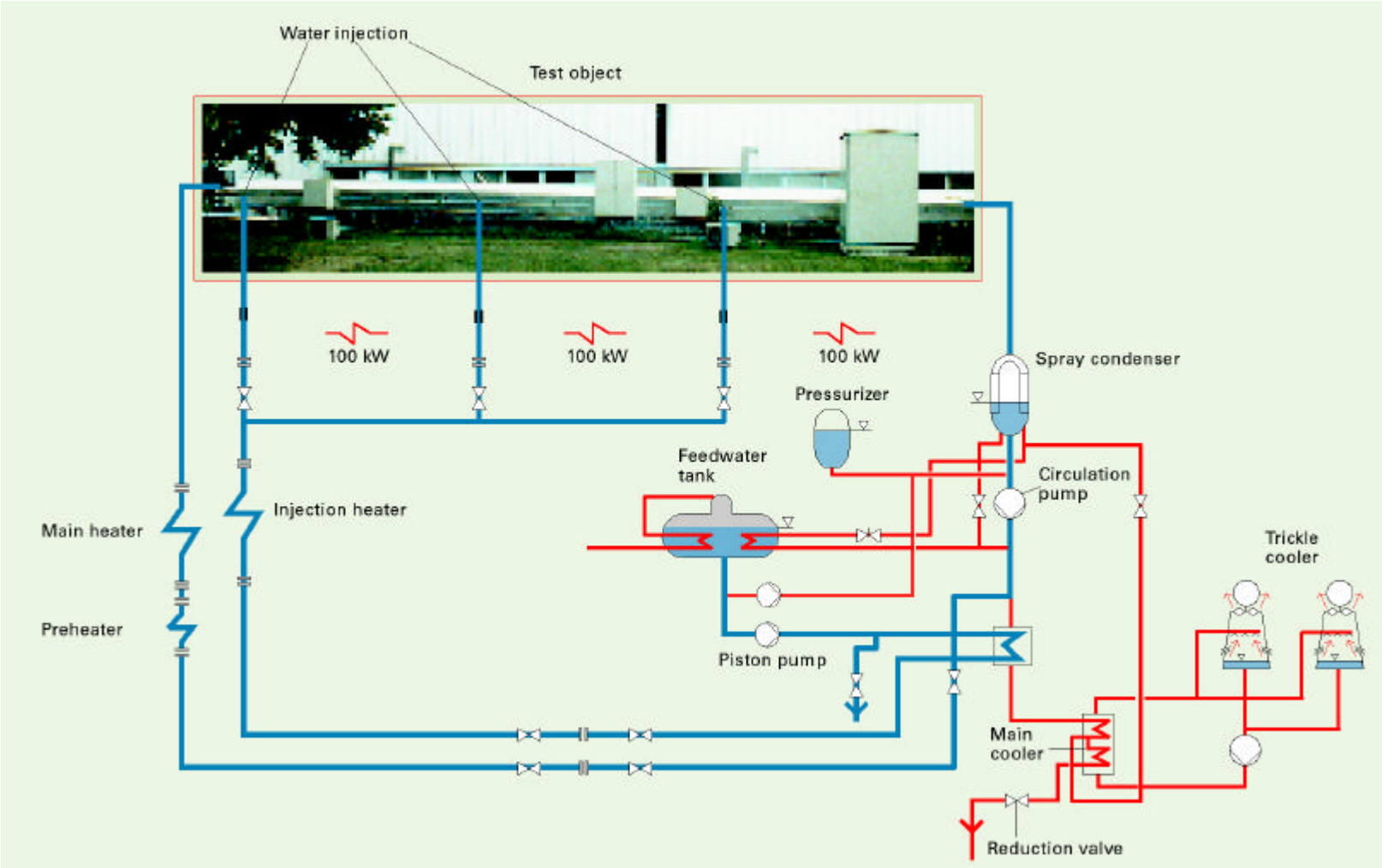
Test-setup



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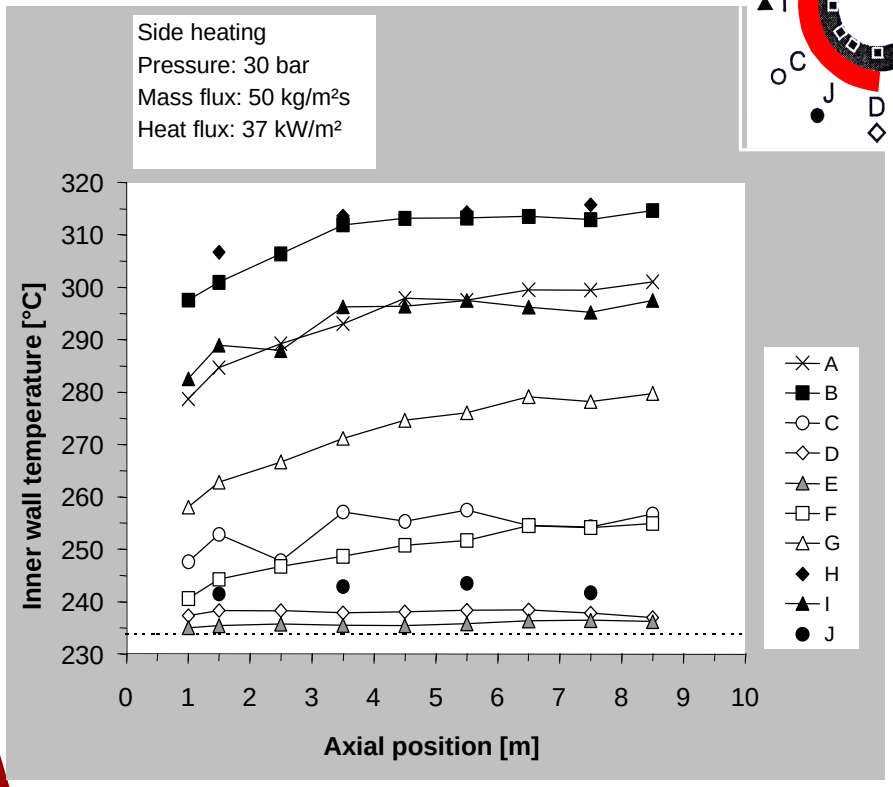
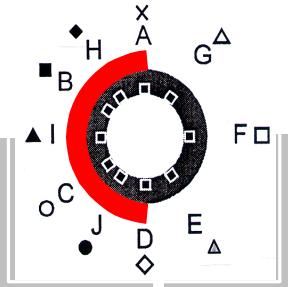
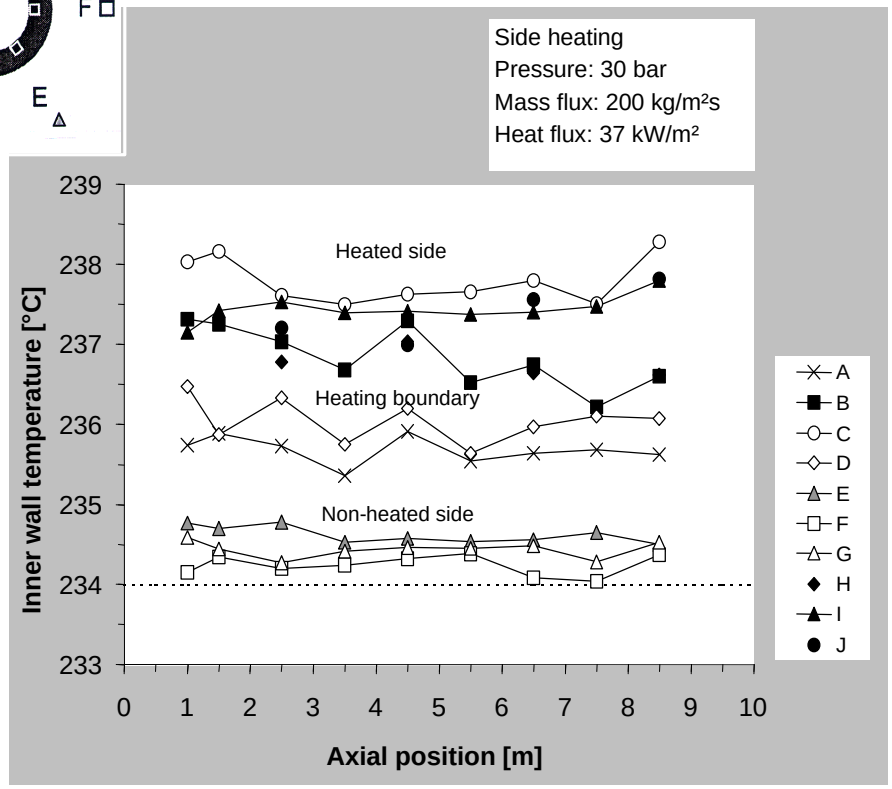
Wall Temperature Distribution as Function of Flow Pattern

Stratified Flow





Annular Flow



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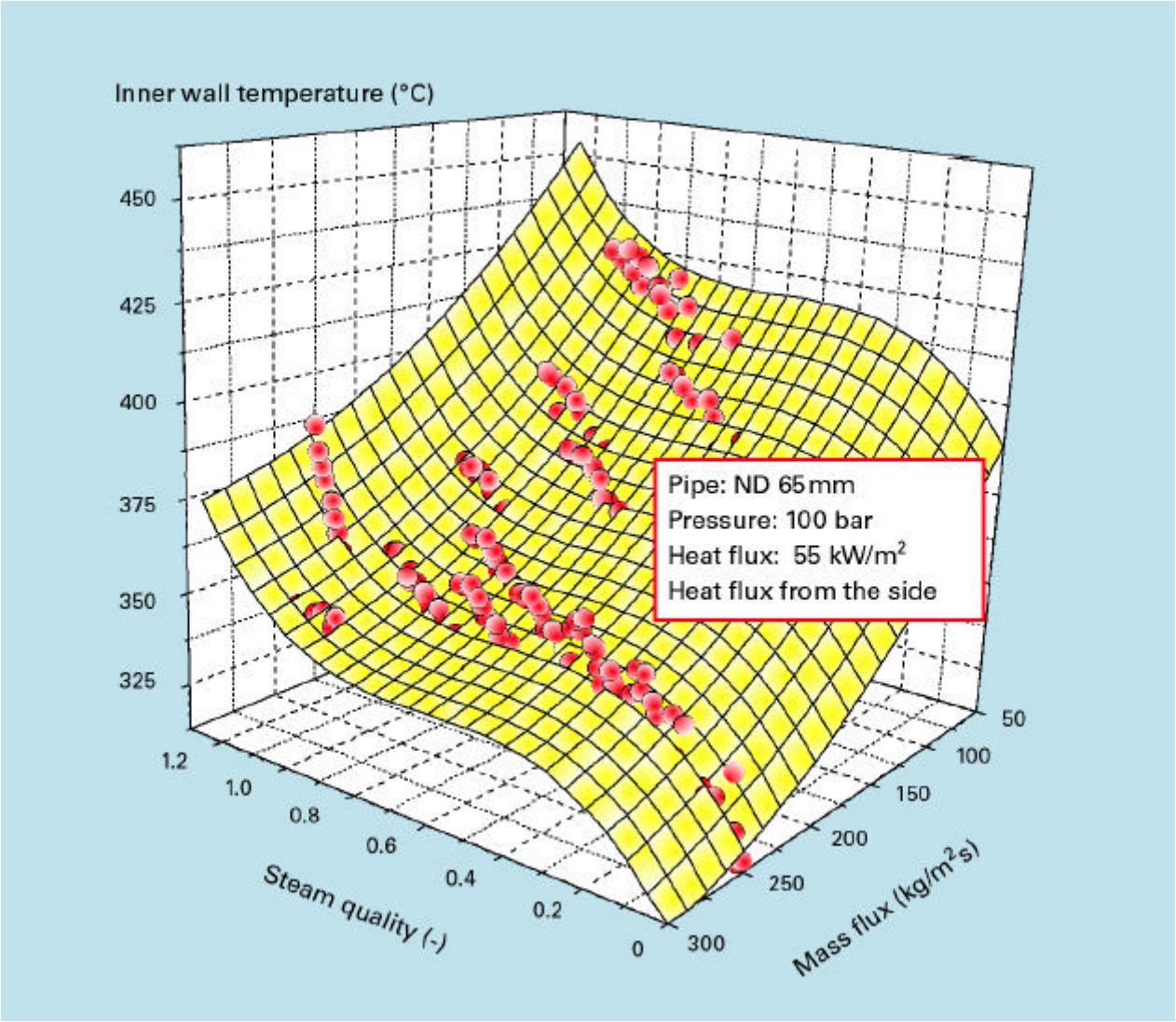
Inner wall temperatures parabolic tough like tubes



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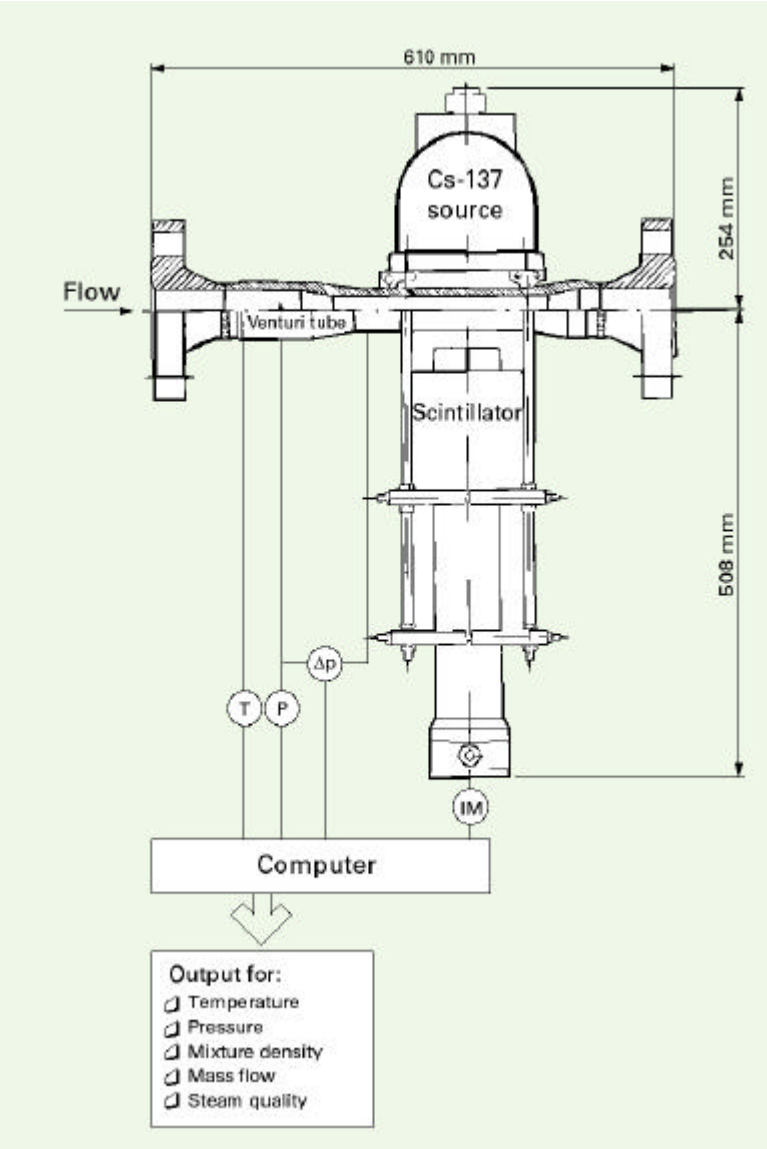
Qualification of two phase flow measurement equipment



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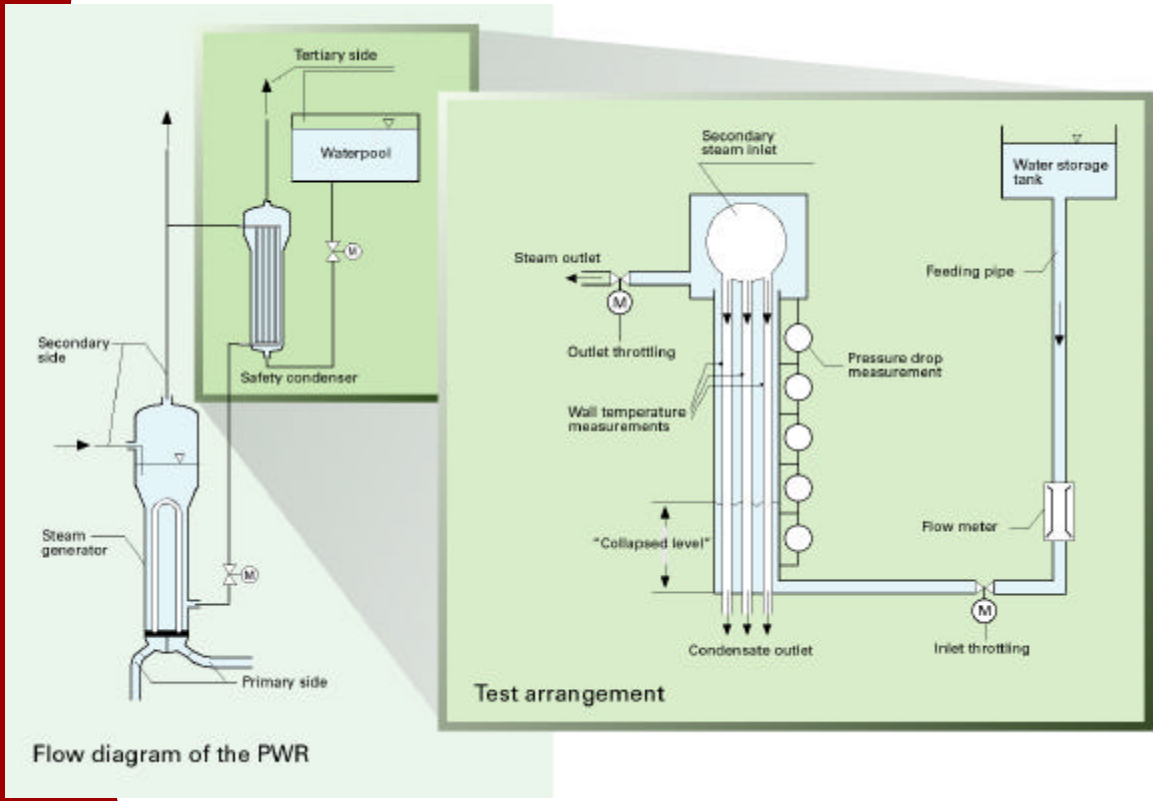
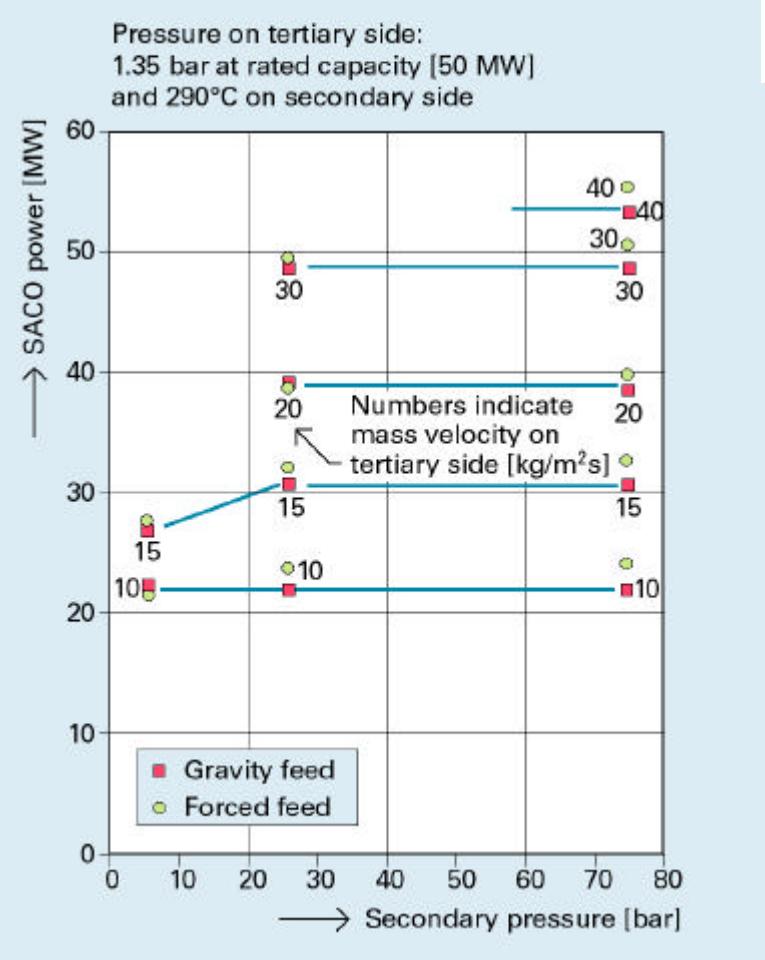
Safety condenser - flow investigations





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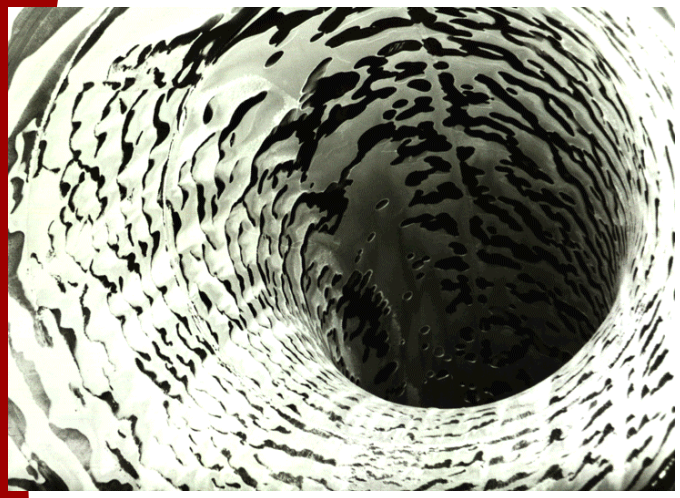
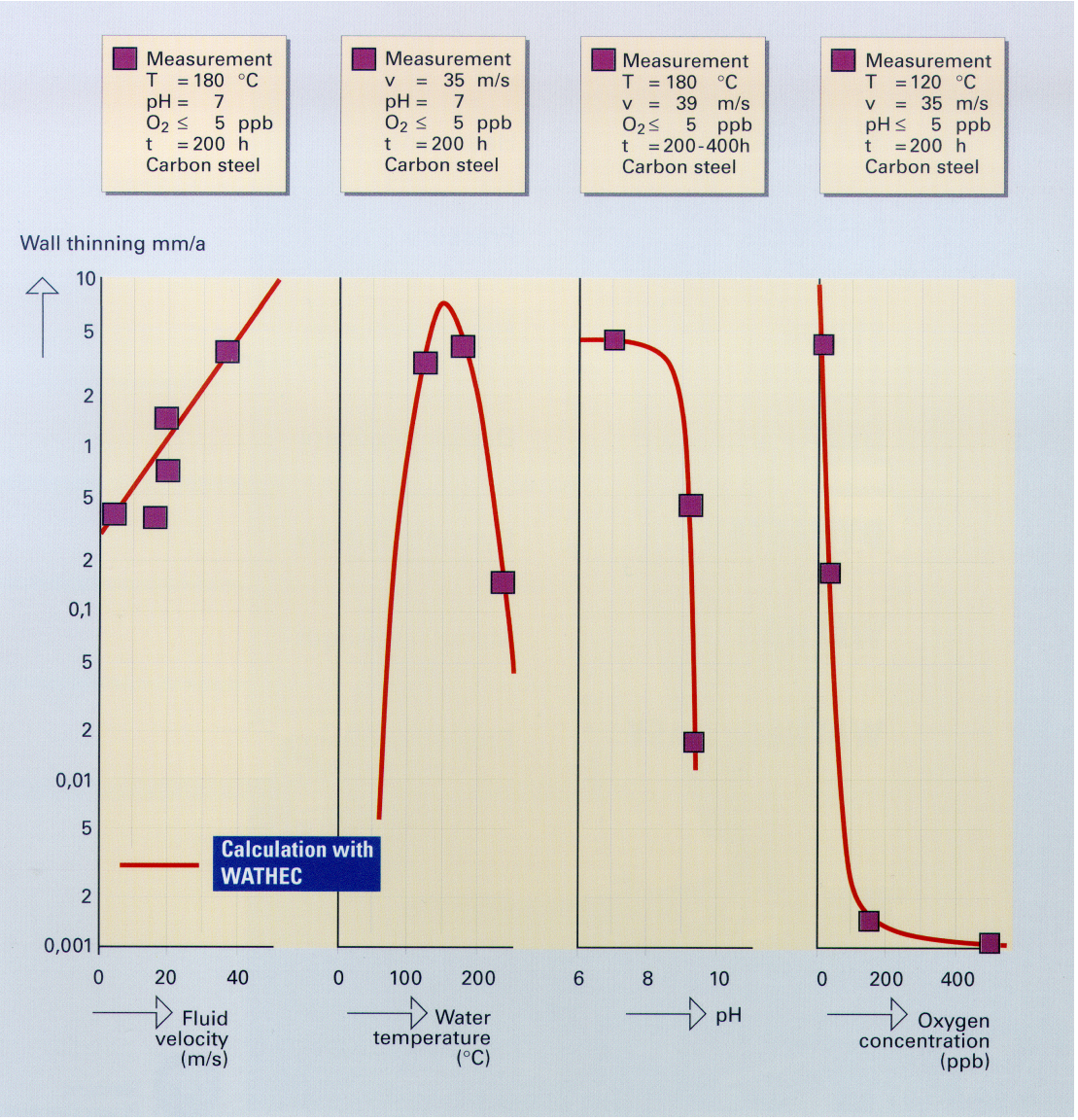
Wall thinning due to erosion corrosion





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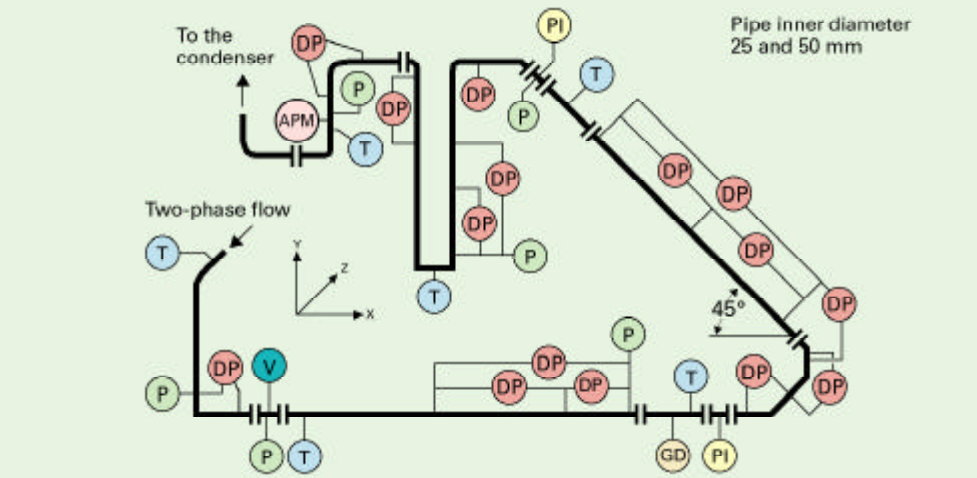
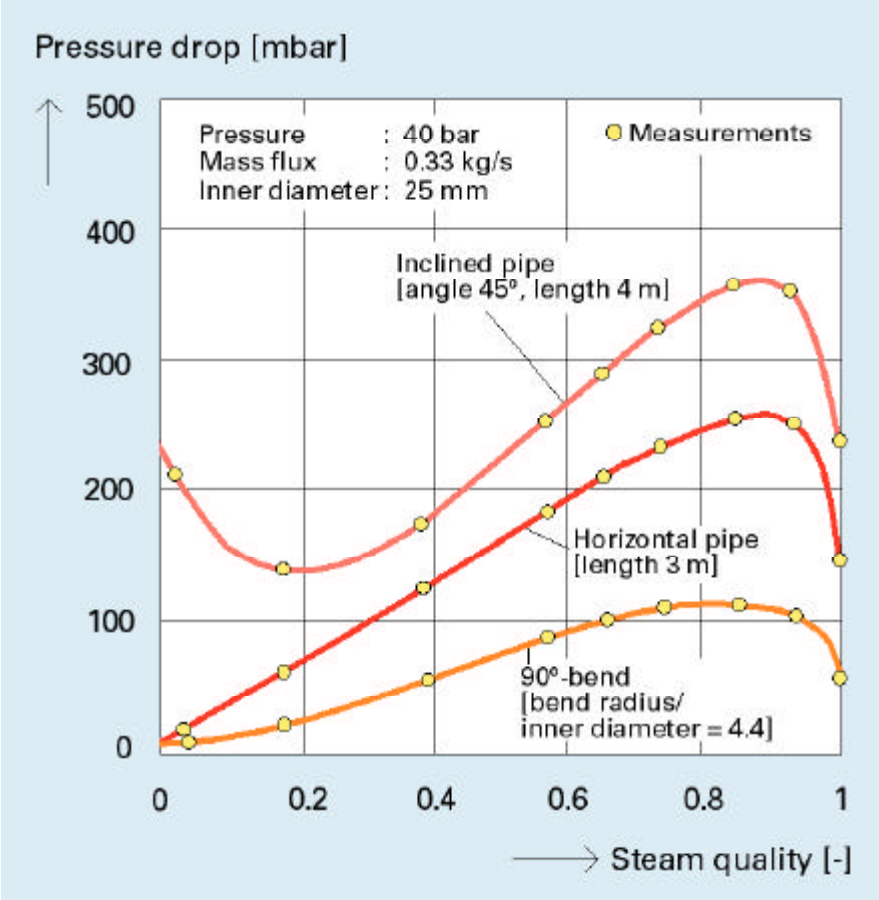
Pressure drops in two-phase flow networks



Testmatrix:

Pressure	$20 \le p \le 100$ bar
Steam quality	$0 \le \dot{x} \le 1$
Mass flux	$100 \le \dot{m} \le 700$ kg/m²s

- P Pressure transducer
- DP Differential pressure transducer
- APM Averaging pitot-meter
- GD Gamma densitometer (5 beams)
- PI Multiple pitot-tube bank
- V Venturi-tube
- T Temperature



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