5. AUXILIARY **SYSTEMS**

Details of most RD-14M auxiliary systems are given in this chapter, however the fast-fill, degas, and electrical systems are given in Appendices D-F, respectively.

5.100 SURGE SYSTEM

The surge system provides three main functions in the primary circuit system:

- it provides a means of controlling the operating pressure,
- it accommodates expansion and contraction of the primary circuit coolant due to temperature and/or phase change, and
- it purges the non-condensable gases from the primary circuit.

Not all CANDU reactors have surge tanks (also referred to as pressurizers). Some reactors use a feed-and-bleed system to perform the functions given above. In RD-14M, the surge tank, shown schematically in Figure 5.1, serves the functions described above only during steady-state operation. Ball valve MV3, located in the line connecting the surge tank to the loop, is normally closed to isolate the surge tank prior to any transient experiments. Thus the RD-14M surge tank is not scaled to its counterpart in the reactor.

The major components of the surge system are described below. The elevation diagram is shown in Figure 5.2.

5.110 Surge System Piping

All surge system piping is ASTM A106, Grade-B carbon-steel pipe. Pipe sizes vary from 0.5-inch (nominal), schedule-80 to 1.5-inch (nominal), schedule-80. All surge system piping is pressure rated to 16.5 MPa (g) at 343°C. Socket-welded joints are used for nominal piping up to 2 inches.

5.120 Surge Tank

Surge tank TK1, shown schematically in Figure 5.1, is a vertical, carbon-steel vessel suspended from the roof of the building. The vessel has a carbon-steel shell (ASME SA106 Gr-C) with a ring-joint flanged closure on the bottom, in which an electric heater is located.

The surge tank has a design pressure of 16.5 MPa (g) at 343°C. The working pressure for the vessel is 13.9 MPa (g) at 337°C.

The empty vessel mass is 2273 kg, and the capacity is 0.549 m^3 .

To enable degassing of the primary system fluid, the surge tank is provided with a degas line to atmosphere regulated by solenoid valve MV1 (see Figure 5.1). The liquid level in the surge tank is determined by measuring the pressure difference between two taps provided near the top and bottom of the vessel, as shown in Figure 5.1. The level is measured with devices associated with device code 5H. Fluid temperature (thermocouple 356T-D1) and flow (turbine flowmeter 15F), to and from the surge tank, are measured in connecting piping. Differential pressure between the inlet header HDR5 and TKI, is measured by differential pressure cell 64Q-DI. The instrumentation is listed in Chapter 6.

Motor-operated valves MV3 and MV19, are both used to isolate the surge system from the primary system.

A bursting disc and a relief valve are used to protect the surge tank from over-pressure. The bursting disc, used in series with the relief valve, protects the relief valve seat from corrosion and simmering by preventing the fluid from contacting the seat of the relief valve. The bursting disc and relief valve are both set at 16.5 MPa (g).

The turbine flowmeter, 15F, is only installed for necessary experiments, otherwise it is removed from the system to prevent damage.

5.130 Surge Tank Heater

A 100-kw immersion heater, HR1, is installed in the bottom of the surge tank via a ring-joint flanged connection. There are 15 U-shaped elements with a total heated length per element ranging from 793 mm to 1095 mm. The maximum height from the flange face to the top of the U-shaped element is 520 mm.

The heater was manufactured by Chromalox, model TMSS-8-1510015MMR Y/8.

The surge tank heater is used to pressurize the surge tank, which in turn pressurizes the primary circuit. Before the surge tank heater is started, the tank water level must be greater than 20%. This ensures there is enough fluid in the surge tank to cover the heater elements. A level recorder, 5H-R1, indicates the surge tank level. Also, valves MV3 and MV19 must be open.

Temperature devices monitor, control, and limit the temperature of the heater and the liquid in the surge tank. Thermocouple 46T protects the heater sheath from over temperature. RTD 45T is used to limit the surge tank liquid temperature. Thermocouple 47T monitors the surge tank liquid temperature.

5.140 Primary-Circuit Pressure Control

The primary pressure is measured at header HDR5 by pressure transducer 12P-D1, and is controlled by the surge tank heater, HR1. The heater is used only to raise or maintain pressure.

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5. AUXILLARY SYSTEMS

The heater is controlled by power controller HR1-PS1, which gets its demand signal from the primary circuit pressure controller 12P-CI, which in turn receives its input signal from 12P-D1. The set-point range is 0 to 15 MPa (g). The surge tank heater is protected from overheating by interlock 46T-KI.

To lower the pressure on the primary side, the set-point on the primary pressure controller, 12P-C1, is lowered, and the vent valve on the surge tank (MV1) is opened remotely by the loop operator. Lowering the pressure is not under automatic control whereas raising (or maintaining) pressure is, as discussed above.

The primary circuit is protected on high pressure by interlock switch 12P-K1. When this set point is exceeded, the surge-tank heater is shut off; both primary pumps, P1 and P2, are shut down; the primary-circuit make-up water pump P4 is shut down; the "PRI pressure high" annunciator is activated; and all four power supplies are shut down. When the primary-circuit pressure falls below the set-point of interlock switch 12P-K2, the "PRI pressure low" alarm is activated and the primary pumps are shut down.

5.150 Surge Tank Level Control

The surge tank is filled by make-up water pump P4 (see Section 5.621 and Figure 5.10), a high pressure, positive-displacement pump. The control signal for the make-up water pump comes from controller 5H-C 1. The controller receives its input signal from the surge tank level, 5H (the level is compensated for density variations), and sends an output signal (to 5H-Cl) to adjust the surge tank level. The normal operating tank level is in the 40 to 50% range with no quality in the primary circuit.

A dual alarm unit, 5H-KI/K2, provides high and low level alarms when the set-points for the surge tank are surpassed. When the surge tank level exceeds the set-point on interlock switch 5H-K1, the "surge TK level high" annunciator is activated, the make-up water pump is shut off, and the primary degassing pump P10 is shut off. When the surge-tank level falls below the set point on the 5H-K2 interlock switch, the "surge TK level low" annunciator is activated, all test section power supplies are shut down, both primary pumps are shut down, and the surge tank heater is shut off.

5.200 DRAIN SYSTEM FOR NATURAL CIRCULATION TESTS

In partial inventory natural circulation tests, two-phase conditions are established by controlled intermittent draining of fluid from a drain connection to inventory tank TK3. The drain system schematic is shown in Figure 5.3. The drain rate, monitored by turbine flowmeter 237F-D1, is controlled manually by adjusting control valve 7H-CV7. For most of these experiments, the drain connection on outlet header HDR7 is used (see Figure 2.24).

Other connections are available. These are as follows:

- 1. Draining from the outlet lines of test sections TS9 (Station 12, Figure 2.9) or TS14 (Station 12, Figure 2.10) (these lines are normally capped).
- 2. Bypassing the inventory tank and venting or draining directly to the vent stack. This can only be done manually.
- 3. Draining from the end cap of inlet header HDR8, to TK3 (see Figure 5.3). A series of steam generator condensation tests (B9801-03) used this drain. No flow control was used for these tests (similar to a blowdown, but with the ability to measure the flow rate and inventory discharged).

The drained fluid is cooled in heat exchanger HX3 before it is collected in the inventory tank. The mass of water collected in this manner is inferred from measurements of the liquid level in the tank.

5.300 BLOWDOWN SYSTEM

Experimental requirements exist to study guillotine breaks in the piping of the primary heat transport system of a reactor. The blowdown system is used to produce the transient conditions that will occur in either an inlet or an outlet header break. The loop coolant is ejected from the loop through a blowdown valve and into the blowdown stack (which transports the steam/water safely to the roof of the RD-14M structure, where it is discharged to the atmosphere). The blowdown line is a 20-inch (nominal) schedule-20 carbon-steel pipe, and was chosen to minimize flow resistance. The blowdown pipe is joined within the loop cabinet to allow for inlet header (HDRS) or outlet header (HDR7) breaks.

Two blowdown valves are available - a fast-acting 6-inch ball valve (approximately 0.25 s opening time) and a fast-acting 2-inch ball valve (approximately 0.10 s opening time) (see Figure 5.4). The valves can be connected to either inlet header HDR8 or outlet header HDR7. A restriction orifice plate in a Grayloc fitting is installed immediately upstream of the ball valve (see Figures 5.4 and 5.5). Various orifice plates are available to simulate a range of break sizes.

A 2-inch drain line, shown in Figure 5.4, is provided to allow condensate (or rain) in the blowdown pipe to flow to the building drainage system.

The blowdown system elevation changes are given in Figure 5.6 (shown with the 6-inch blowdown valve, MV8).

5.400 LOOP INSULATION

The RD-14M loop is lagged using light-weight (200 kg/m^3) , low thermal-conductivity (0.91 W/m.K) hydrous calcium silicate pipe insulation, covered with canvas lagging cloth and/or aluminum sheet. This insulation is 65-mm thick for nominal pipe sizes from 2 to 3.5 inches, and 76-mm thick for 1-, 1.25-, and 4-inch pipe. Non-pipe items such as the headers, steam generators

shell and plena, and pump volutes, are also well insulated. The heated sections are insulated with granular vermiculite fill as described in Section 2.240.

5.500 TRACE HEATING

Heat losses from feeders can be significant, in particular for low-flow, low-power tests (e.g., natural circulation tests). Heat losses in RD-14M have been characterized (see Section 8.700).

To counteract the feeder heat losses, the feeders and end-fittings are trace-heated with fiberglass "HOTFOIL" trace heating tapes (see Figure 5.7).

To guard against adding heat into the system, four variable transformers are installed and adjusted for the heat losses expected. Each transformer controls the power to each bank of inlet and outlet piping, one control for each set of five feeders. Four meters are installed to indicate the power the tracer heaters use. Figure 5.7 also shows temperatures at the inlet and outlet of the test section and near the headers. These are the thermostat set-points. If the temperature of the thermostat is below the set-point, the trace heating is on, if below the set-point it switches off (automatically). There is also a remote shutoff switch to shut the system down. Nominal full power trace heating powers (as per design specifications) for each feeder are given in Table *5.1.* The total trace heating power is approximately 10 kW/pass to the inlet feeders and 12 kW/pass to the outlet feeders. The electrical schematic for the trace heating system is given in Appendix F.

In 1999 January, the trace heating on TS 11 outlet feeder was replaced with three "ACCUTRON" mineral insulated trace-heating cables, installed at 120° intervals around the pipe circumference (see Figure 5.8). These cables were attached and cemented to the pipe (as per manufacturer's requirements). The power to these heating cables has the capability of being adjusted from 0-10 kW.

5.600 MAKE-UP WATER ADDITION

There are two systems available to add make-up water to the loop. They are referred to here as the metered and non-metered systems.

5.610 Metered Make-Up Water Addition System

Some experiments require the primary system to be injected with known quantities of water. This installation was completed in 1994 and consists of an inventory tank, TK4, and a level transmitter, 12H. The injected water quantities are inferred from 12H. The degassing pumps P10/P11 are used to inject the primary system through the ECI lines to the headers with water from tank TK4. The tank is open to atmosphere, made from stainless steel, and has a capacity of approximately 0.7 m^3 . The metered make-up water addition system schematic is given in Figure 5.9.

5.620 Non-Metered Make-Up Water Addition System

The other make-up water system could be used to replace any losses in the primary or secondary system due to leaks during normal operations (this system is isolated during partial inventory tests and is not typically used for other tests either). The amount of inventory injected into the primary and/or secondary systems is not measured.

The water is supplied to the loop, via a 3-inch line, from a distilled water system. This line also services the fast-fill system (see Appendix D) when the loop is shutdown and depressurized. Positive displacement pumps are used to fill the loop because the distilled water supply is at a lower pressure than RD-14M. Check valves, in the loop piping, protects the building's distilled water system from over-pressure and reverse flow. Relief valves are installed on the discharge piping of the pumps to protect the piping system from over-pressure. The non-metered make-up water addition system schematic is shown in Figure 5.10.

5.621 Primary System

Make-up water is injected into the primary system upstream of pump P2. A Milton Roy positive displacement pump, P4, is used for this purpose. The surge tank level controller, associated with the 5H level measurement (compensated for temperature), is used to determine the primary system water level. The pump is shut off when the desired operating level is reached.

5.622 Secondary System

Make-up water is injected into the secondary system piping downstream of heat exchanger HX2. A Milton Roy positive displacement pump, **P5,** is used for this purpose. The jet condenser level controller, associated with the 4H level measurement (compensated for temperature), is used to determine the secondary system water level. The pump is shut off when the desired operating level is reached.

5.700 EVENT SEQUENCER

A Durant Systems model 6450 programmable timer is used for precise event sequencing during an experiment. This versatile controller can time and energize or de-energize up to 16 channels in any selected sequence, according to a set of instructions given by the loop operator. These channels are used for such things as activating/deactivating valves (open/close), starting pump rundowns, and starting the ECI pumps.

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5. AUXILLARY SYSTEMS

TABLE 5.1

NOMINAL FULL-POWER TRACE HEATING VALUES

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FIGURE 5.1: Surge Tank (TK1) Schematic

FIGURE 5.2: Surge Piping System Schematic/Elevation Diagram

FIGURE 5.2 Legend

FIGURE 5.3: Drain System Schematic

FIGURE 5.4: Blowdown Valves

FIGURE 5.5: Blowdown Orifice

FIGURE 5.6: Blowdown Line Elevation Diagram

FIGURE **5.7:** Trace Heating Arrangement

FIGURE **5.9:** Metered Make-Up Water Addition System Schematic

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6. INSTRUMENTATION AND DATA ACQUISITION

6. INSTRUMENTATION AND DATA ACQUISITION

6.100 INSTRUMENTATION

6.110 Introduction

RD-14M is extensively instrumented to measure parameters such as temperatures, pressures, flows, levels, and voids (gamma densitometers and conductivity probes). Each instrument in RD-14M is hard-wired into a panel near the data-acquisition system. A subset of available instruments is sampled during an experiment. The experimenter can choose the instruments to be recorded by the data-acquisition system for a particular experiment. This is done in software by providing a scan list of instruments to be sampled. There are a few instruments used for alarm, control, and trip functions that are recorded for all experiments.

All instruments provide a proportional 0-5 V, 1-5 V, or **±** 5 V output signal to the data acquisition system. All measurements involve a series of components, typically a sensing element, a transmitter, cables, a signal conditioner, and an amplifier.

Tables 6.1-6.3 are lists that summarize RD-14M instruments on the primary side, secondary side, and ECI system, respectively, and are sorted by measurement type (e.g., temperature, pressure, flow). Table 6.4 contains miscellaneous instruments.

Table 6.1 does not include the FES thermocouples. Also, the primary-side instruments in the table that do not have a device type are instruments that have not been used in any tests to date (the instruments are likely not installed but taps are available).

Tables 6.1-6.4 list the following:

- Device Code: a unique alpha-numeric identifier assigned to a specific instrument location according to predetermined codes, as specified on construction blueprints (an exception to this uniqueness is the FES thermocouples - see Section 6.122).
- Device Type: an identifier assigned to describe the type of RD-14M instrument (in some cases it identifies the manufacturer) and components used for the measurement.
- Measurement Location: keywords or abbreviations used to describe the measurement location of an instrument.
- Reference: Figure and Station Numbers from diagrams found in other sections of this document, or a specific engineering drawing, found in Appendix B. The station number identifies the location of the instrument in the corresponding figure.

Table 6.5 summarizes the different types of instruments available in RD-14M. Each of these is described in the sections to follow.

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6. INSTRUMENTATION AND DATA ACQUISITION

6.120 Temperature Measurements

Temperature measurements are made using Type-K thermocouples (Nickel - 10% Chromium (+) and Nickel - 5% Chromium (-)), or resistance temperature devices (RTD's). Type-K thermocouples can be used over the range -200 to 1250'C, but are more commonly used for the 0 to 1000°C range where the linearity is better. Even over this range, Type-K thermocouples are non-linear. If the thermocouple conditioning circuitry does not correct for non-linearity, then a correction should be applied to the indicated temperature (this will be included in the uncertainty analysis report). This is the case for temperature measurements made with Ectron and Bailey amplifiers/transmitters. Thermocouple channels using Ronan and Promac amplifiers/transmitters automatically correct for the non-linearity of the Type-K thermocouple in its hardware. These devices are summarized in Table 6.5.

6.121 Fluid Temperature Measurements

The thermocouples used in fluid temperature measurements are sheathed in stainless-steel closed-end tubes (shields) to prevent corrosion and mechanical damage from high-temperature steam/water flows. The thermocouples are designed such that the actual junction is in the tip of the protective thin-walled shield and has a very low thermal inertia. Thermocouples measuring fluid temperature are typically mounted such that the tip of the protective shield is at the centerline of the piping component.

6.122 FES Sheath Temperature Measurements

The FES sheath temperatures are measured with factory-made thermocouples that were installed on the inside of the heating-element sheath, and then contacted to the inside surface of the sheath using a swedging technique that was developed in-house. The thermocouples are imbedded in the MgO annulus layer between the inside heater element and the outside sheath (see Figure 2.17).

All instrument locations are identified uniquely by their device code except for the FES thermocouples. If a thermocouple fails and cannot be repaired, it is taken out of service and the junction (wiring) used for the failed thermocouple is switched to another FES thermocouple. There are more thermocouples available on an **FES** than are typically recorded for any given test. The device code stays with that junction, so the new **FES** thermocouple location takes on the device code of the failed FES thermocouple.

6.123 Surface Temperature Measurements

Surface temperatures can be measured using 1/16-inch, K-type thermocouples (with Ronan transmitters) strapped to the outside surface with a metallic hose-clamp. These thermocouples are encased in Thermon (Grade T-63), a high-temperature heat transfer cement, and wrapped with insulation to ensure accurate outer wall temperature measurements.

6. INSTRUMENTATION AND DATA ACQUISITION

6.124 Thermocouple Transmitters/Amplifiers

As previously mentioned, there are four types of thermocouple transmitters/amplifiers used in RD-14M: Bailey, Ectron, Ronan, and Promac. The respective RD-14M device types and manufacturer model numbers are given in Table 6.5.

The Bailey solid-state transmitters/amplifiers are installed in the RD-14M control room and are connected to thermocouples via thermocouple extension wire. These transmitters/amplifiers do not have cold-junction compensation, nor is the output corrected for non-linearity in the Type-K thermocouple. Cold-junction compensation is provided by an Acromag Model 350 Reference junction.

The Ectron solid-state transmitters/amplifiers are installed in the RD-14M control room and are also connected to thermocouples via thermocouple extension wire. These transmitters/amplifiers have integral cold-junction compensation, however the output is not corrected for non-linearity in the Type-K thermocouple.

The Ronan solid-state transmitters/amplifiers are located in cabinets throughout the experimental loop. These transmitters/amplifiers have integral cold-junction compensation, and the output is corrected for non-linearity in the Type-K thermocouple. The current non-linearity correction is for a span of 0-1000°C. In some previous tests, where a span of **100-350'C** was used, a different non-linearity correction is to be applied [6.1].

The Promac solid-state transmitters/amplifiers are located in cabinets throughout the experimental loop. These transmitters/amplifiers also have integral cold junction compensation and the output is corrected for non-linearity in the Type-K thermocouple. The current non linearity correction is for a span of 0-350°C. The non-linearity correction is obtained by adjusting the midpoint to the proper value with a potentiometer during the calibration procedure.

Calibrations for the thermocouple transmitters and amplifiers described above are carried out using the following procedures:

- 1) The output from the thermocouple amplifier/transmitter (normally connected to the data acquisition system) is connected to an accurate digital voltmeter in the RD-14M control room via a plug-in jumper cable.
- 2) The thermocouple is then physically disconnected from the amplifier/transmitter, and a hand-held thermocouple calibrator is connected in its place. This calibrator is essentially an accurate millivolt source that mimics the response of a Type-K thermocouple.
- 3) The calibrator's millivolt signal is set to the value corresponding to 0° C, and then set to the full scale value (either 350 or 1000°C), and the amplifier/transmitter output

voltage is read. If necessary, the amplifier/transmitter zero and span are adjusted to give the desired output voltages at 0°C and at full scale.

- 4) The output voltage from the thermocouple/conditioning circuitry is recorded. This measured voltage can then be compared with that from previous calibrations to check for calibration drift.
- 5) The calibrator is set for five intermediate temperatures and the amplifier/transmitter output voltages are recorded.

The above procedure calibrates the thermocouple conditioning circuitry. The actual thermocouple junction itself is not calibrated.

6.125 RTD Temperature Measurements

The Resistance Temperature Devices (RTDs) are used for temperature measurements whenever accuracy in excess of that provided with thermocouples is required. The zero and span set-points for an RTD are more stable than those of thermocouples, and the device is more linear (as with the thermocouples, accuracy can be increased by applying a non-linearity correction).

Problems with RTDs are usually related to the sensor itself. The platinum wire sensor is physically larger than a thermocouple junction and is wound over the length of some sort of support. The measurement is sensitive to any sharp temperature gradient along this length. Also, the thermal response of an RTD is slow (compared with a thermocouple), due to poor contact of the platinum sensor wire with the measuring point and the large thermal inertia of the protective sheath. The problems associated with RTD temperature measurements in RD-14M have led to many of them being replaced with thermocouples over the years.

Two different types of RTD transmitters have been used, as indicated in Table 6.5.

The RTD sensors/transmitters are calibrated as a unit. The RTD is physically removed from the loop and calibrated against a standard (reference) RTD by immersing both RTDs in a temperature controlled salt bath.

6.130 Pressure and Differential Pressure Measurements

Gauge and differential pressure measurements are carried out in RD-14M using strain gauge and capacitance-type pressure transducers.

Pressure cells connect to the loop components via liquid-filled sense lines. All pressure cells in the loop are located below the points of measurement to ensure that sensing lines are always full during a test. For gauge pressure measurements, the shortest possible horizontal sense line is used. The sense lines for differential pressure cells include horizontal legs at the loop end to prevent emptying of the vertical portion of sense lines. The sense lines are arranged and filled so that there is zero head at the cell.

Before each experiment, the zero offset voltages of all differential pressure cells are checked. This is done with primary-side loop conditions of 2 MPa, zero flow, and fluid at ambient temperature. If the offset is greater than the accuracy of the cell (e.g., 0.25% FS), then the cell is re-zeroed.

6.131 Strain Gauge Transducers/Transmitters

Table 6.5 summarizes the different types of strain gauge pressure transducers/transmitters used in RD-14M.

The BLH, Dynisco, and Transducer Inc. transducers/transmitters (with Ectron amplifiers) have a foil strain gauge. Operational experience has shown that these instruments are prone to zero drift due to the foils having residual memory.

Druck transmitters have a silicon-based stain gauge that has no zero drift and no residual memory. Output from the transmitter is a 4 to 20 mA signal that is proportional to the applied gauge pressure.

The pressure range of each strain gauge transducer is calibrated with the cell, wiring, and amplifiers as a unit, against a pressure standard reference source (EATON, Druck, or Fluke). The reference source is connected to the cell and the output voltage of the cell is recorded as a function of the reference pressure. The zero and span of the cell are adjusted to give the desired output voltages (i.e., 1-5 V), corresponding to the range of the cell.

6.132 Capacitance-Type Pressure Transmitters

Table 6.5 summarizes the different types of capacitive pressure transmitters used in RD-14M. The two types are the Rosemount model 1151 series and model 3051 series. Both of these cell types have a variable-capacitance sense element. The differential capacitance between the sensing diaphragm and fixed capacitor plates is electronically converted to a 4 to 20 mA signal that is proportional to the applied gauge or differential pressure. This current is applied to a precision resistor to produce a suitable voltage signal that is measured by the data-acquisition system.

There have been three types of Rosemount 1151s installed in RD-14M. The original *115* **1s** installed were analogue cells (these were pressure cells previously installed in RD-14). New Rosemount pressure transducers initially installed in RD-14M were first generation smart cells, then later on second generation smart cells were installed. Many of the analogue cells have since been converted to second generation Rosemount pressure cells, by using a conversion kit that only replaced the electronic components of the cell, and not the cell itself. The RD-14M calibration database, part of the COG Safety Thermalhydraulics database [6.2], has a field that indicates the type of cell installed.

The Rosemount 3051 is a newer generation cell designed to replace the 1151. The 3051 is a smart cell, and has a greater accuracy and is less susceptible to drift due to ambient temperature change than the 1151. Rosemount 3051 differential pressure transmitters also have reduced sensitivity to span and zero shift with static pressure when compared to the 1151 transmitters.

For calibration of the Rosemount transmitters, the instrument is removed from its location in the experimental facility and calibrated against a standard calibrated (reference) test cell.

For all smart cells, a microprocessor controls transmitter operation so that the transmitter calibration is not dependent on range. The transmitter is initially calibrated over the maximum possible range. Following calibration, the transmitter range can be changed, within limits, to be a fraction of the calibrated range. Zero and span adjustments can be made after installation without affecting the calibration.

6.140 Flow Measurement

Flow is measured or calculated using various devices, as summarized in Table 6.5. The different devices are turbine, vortex, and orifice flowmeters.

6.141 Turbine Flowmeters

Turbine flowmeters (TFMs) are designed and calibrated for single-phase liquid flow. Both unidirectional and bidirectional TFMs are in service in the loop. There are two manufacturers of TFMs used in RD-14M, as indicated in Table 6.5. It should be noted that turbine flowmeters only provide qualitative measurements under two-phase steam/water flow conditions.

Some of the features of these instruments are they are operable with fluid temperatures up to 400'C, have high overrange capability, and can be operated bidirectionally. The advantages of turbine flowmeters in the RD-14M configuration are accuracy, repeatability, extended flow range, rapid response time, and retention of calibration.

In RD-14M the availability and performance of turbine flowmeters is strongly affected by rotor and bearing integrity. Meters must be regularly reconditioned by replacing the bearings, and in some cases, the rotor. A discussion of turbine flowmeter performance is provided in Section 8.200.

For calibrations, turbine flowmeters are removed from the loop and calibrated individually against a master turbine flowmeter located in the RD-14M Flow Calibration Facility.

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6.142 Mass Flowmeters

Mass flows are calculated for the boiler feedwater by a mass flow computer using readouts from turbine flowmeters installed on each boiler feedwater line. The density is derived from a temperature measurement, assuming saturation conditions. Values for the flow coefficient used in the computer are experimentally determined from regular calibrations of the TFMs in the RD- 14M Flow Calibration Facility.

6.143 Vortex Flowmeters

Vortex flowmeters were originally installed on the four ECI injection lines, but were replaced by turbine flowmeters as the accuracy of the vortex meters was affected by vibrations caused by the ECI pumps. The only remaining vortex flowmeter in service in RD-14M is situated on the ECI pump P14 bypass line. Vortex flowmeters are unidirectional.

6.144 Orifice Flowmeters

The boiler steam mass flows are calculated by the steam mass flow computer using the differential pressure across an orifice and the density derived from a temperature measurement, assuming saturation conditions. Values for the flow coefficient used in the computer are experimentally determined from calibrations of the orifice in the RD-14M Flow Calibration Facility.

Mass flow rates are not calculated, but can be, for the orifices located on the boiler downcomer lines and the header interconnects. The mass flow rates may be calculated using the equation for sharp edged orifices.

6.150 Level Measurements

All level measurements are made using Rosemount model 1151 series differential pressure cells, with each ΔP measurement density compensated to indicate the actual fluid level in a vessel. Calibrations are performed on the ΔP cells, as well as the density compensators, on a regular basis.

There are some ΔP cells on loop inventory tanks that are calibrated directly for mass. The calibrations are obtained by draining fluid from the vessel and calibrating the weight of fluid versus the AP cell voltage output.

6.160 Void Measurements

Void is inferred from two types of devices (summarized in Table 6.5): gamma densitometers and conductivity probes.

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6.161 Gamma Densitometers

The RD-14M gamma densitometers consist of **I** to 3 collimated beams from a radioactive Cesium source, which is attenuated by passage through pipes in the RD-14M loop containing process water. Each beam has a detector mounted on the opposite side of the pipe. The attenuated beam changes the current passing through an ionized gas and the current is converted to a 0-5 V output signal, with each beam having a separate output. The current signals are related to the density or void fraction of the steam/water mixture in the pipe.

6.162 Conductivity Probes

Conductivity probes have been installed in various locations of RD-14M to qualitatively detect void. These instruments were developed in-house.

6.170 Power

Power is calculated by two systems, as summarized in Table 6.5.

6.171 Power Supply Controllers and Power Meters

One system that provides a power measurement relies on direct measurement of the voltage and current supplied by a power supply. For each power supply, measurements of the supplied voltage and current are fed into a Wattmeter (power card). The Wattmeter provides a voltage signal output that is proportional to the actual power output of the power supply. This voltage signal is then fed into the data-acquisition system, thus providing a direct measurement of the power (on a per power supply basis only).

The voltage output of the Wattmeter is also fed into the corresponding power-supply controller located in the RD-14M control room. This is the feedback signal that the controller uses in controlling the power-supply output.

The power measurements provided by this system are less accurate than those provided by the Fluke system, which is discussed in the next section. However, the Fluke system is substantially slower than the direct power measurements. For this reason, the powers measured by the Fluke system should be used under steady-state power conditions. During transient changes in power, the direct measurement of power is the only system capable of recording the transient.

Isolation amplifiers were installed between the outputs of the Wattmeters (power cards) and the data-acquisition system in 1997 to reduce line noise [6.3]. Prior to the addition of the isolation amplifiers, line noise occurred in the voltage signal from the Wattmeters, which manifested itself as noise in the measured power of the power supplies.

6. INSTRUMENTATION AND DATA ACQUISITION

6.172 Fluke System

Power is calculated for each heated section from true RMS voltage and current measurements which are fed into the Fluke computer. Voltage is measured across each heated section, with the voltage taps for these measurements located on the busses at the ends of the heated sections where power is distributed to the seven fuel element simulators. Current is measured by thermal RMS meters, utilizing a current shunt to produce a voltage which is then fed to the Fluke computer. Power values are calculated by the Fluke computer, and an average of ten measurements, sampled over a period of 22 seconds, is used for each data point. The averaged values are written to a data file. While the Fluke computer provides accurate power readings, the measurement is slow due to the period required for data collection and averaging. To measure and record the powers for each of ten heated sections requires a period of 220 seconds.

The Fluke system is a stand-alone system, separate from the main data-acquisition system, and generates it's own data file (synchronized to the main data-acquisition system).

6.180 Measurements for Pumps

The primary pumps of the loop are instrumented to provide current, speed and torque measurements. The secondary pumps are instrumented to measure speed.

6.200 DATA ACQUISITION SYSTEMS

6.210 Main System

The main data-acquisition system consists of a Digital Equipment Corporation MicroVAX II computer, an input multiplexer, and a number of 12-bit analogue-to-digital converters. The data are stored on a disk during acquisition and later transferred to permanent storage devices.

The maximum number of input channels is 768, and the maximum scan rate with 600 channels of data collection is approximately 50 ms/scan (20 Hz), based on past loop operational experience.

6.220 Auxiliary Systems

Each data channel that feeds into the main system multiplexer is also routed to a jack panel. The jack panel allows for the connection of high-input-impedance analogue devices, such as digital voltmeters and recorders, to desired data channels for simultaneous (independent) measurement. A two-channel Hewlett Packard X-Y recorder and three digital voltmeters are permanently mounted near the jack panel for this purpose. Other portable instruments, such as oscilloscopes, can also be used.

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6. INSTRUMENTATION AND DATA ACQUISITION

The jack panel also allows for the connection of auxiliary data-acquisition systems. Three auxiliary data-acquisition systems have been used:

- 16-channel Vaxstation This system allows for the display of up to 16 data channels. No data are actually recorded.
- 48-channel personal computer system This system allows for the graphical display of up to 48 data channels.
- High-speed data-acquisition system This system is personal computer (PC) based, and allows for the high-speed acquisition (over 100 Hz) of up to 64 data channels. Data are stored locally on the PC. This system can be started (triggered) by the main data-acquisition system, and can be accurately time synchronized with the main data acquisition system.

REFERENCES

- [6.1] Melnyk, A.J., "Effect of Non-Linear Behaviour of Thermocouples on RD-14M Measurements," memorandum to COG Working Party #5 members, THB-91-497, 1991 November 25.
- [6.2] Swartz, R.S., "Migration of the FoxPro Database to MS-Access," memorandum to COG Working Party #5 members, STHB-99-053, 1999 April 12.
- [6.3] Sanderson, T.V., Melnyk, A.J., and Ingham, P.J., "Improvements to RD-14M Power Measuring System," memorandum to COG Working Party #5 members, STHB-97-159, 1997 June 21.

TABLE **6.1**

SUMMARY OF PRIMARY-SIDE INSTRUMENTS

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6. INSTRUMENTATION AND DATA ACQUISITION

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TEMPERATURE RD-14M RD-14M Reference Device Code | Device Type | Measurement Location | Figure No. | Station No. 2T-D14 KTC-R B2-TB4-S25 2.29 n/a 2T-D15 | KTC-R | B2-TB4-S12 | 2.29 | n/a 2T-D16 | KTC-R | B2-TB4-S50 | 2.29 | n/a 2T-D21 | KTC-R | B2-TB8-S12 | 2.29 | n/a $2T-D22$ KTC-B $|B2-TB8-S50$ $|2.29|n/a$ 2T-D23 KTC-B $| B2-TB8-S84 | 2.29 | n/a$ 2T-D24 | KTC-R | B2-TB8-S92 | 2.29 | n/a 2T-D25 | KTC-R | B2-TE1-S37 | 2.29 | n/a 2T-D26 | KTC-R | B2-TE1-S25 | 2.29 | n/a 2T-D27 | KTC-B | B2-TE1-S12 | 2.29 | n/a 2T-D29 | KTC-B | B2-TE1-S84 | 2.29 | n/a 2T-D30 KTC-B $| B2-TE1-S92 | 2.29 | n/a$ 2T-D32 KTC-R $| B2-TE6-S25 | 2.29 | n/a$ 2T-D34 | KTC-R | B2-TE6-S50 | 2.29 | n/a 2T-D36 | KTC-R | B2-TE6-S92 | 2.29 | n/a 1T-D28 $|$ BOI T $|_{2.29}$ $|_{\text{n/a}}$ 2T-D17 $|$ BO2 T $|_{2.29}$ $|_{\text{n/a}}$ 2T-D18 $|BO2T|$ 2.29 $|n/a$ 2T-D19 $|BO2 T | 2.29 | n/a$ 2T-D20 \vert BO2 T \vert 2.29 \vert n/a 2T-D28 \vert BO2 T \vert 2.29 \vert n/a 2T-D31 \vert BO2 T \vert 2.29 \vert n/a 2T-D33 $|$ BO2 T $|_{2.29}$ $|_{\text{n/a}}$ 2T-D35 $|$ BO2 T $|_{2.29}$ $|_{\text{n/a}}$ 1TFB | KTC-P | HS11 FDR OSRF | 7.1 | 18 1TFT $|$ KTC-P $|$ HS11 FDR OSRF $|$ 7.1 $|$ 18 2TFB | KTC-P | HS11 FDR OSRF | 7.1 | 17 2TFT | KTC-P | HS11 FDR OSRF | 7.1 | 17 3TFB | KTC-P | HS11 FDR OSRF | 7.1 | 16 3TFT | KTC-P | HS11 FDR OSRF | 7.1 | 16 $4TFVL$ $|KTC-P|$ $|HS11 FDR OSRF|$ $|7.1|$ $|13|$ 4 TFVR | KTC-P | HS 11 FDR OSRF | 7.1 | 13 $5TFS$ $|KTC-P|$ $|HS11 FDR OSRF|$ $|7.1|$ $|11|$ 5TFT | KTC-P | HS11 FDR OSRF | 7.1 | 11 6TFVL KTC-P HS **1I** FDR OSRF 7.1 9 6TFVR KTC-P **HSI I** FDR OSRF 7.1 9 7TFB | KTC-P | HS11 FDR OSRF | 7.1 | 6 7TFT | KTC-P | HS11 FDR OSRF | 7.1 | 6 8TFB | KTC-P | HS11 FDR OSRF | 7.1 | 3

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 $8TFT$ $|KTC-P|$ $|HS11 FDR OSRF|$ $|7.1|$ $|3$

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6. INSTRUMENTATION AND DATA ACQUISITION

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6. INSTRUMENTATION AND DATA ACQUISITION

¹ These devices have used both KTC-B and KTC-R device types.
² These devices have used KTC-B, KTC-R, and RTD device types.
³ Modified to measure HS13 outlet feeder wall temperature for B96 tests

⁴ Modified to measure HS13 inlet feeder wall temperature for B96 tests

⁵Installed on bottom instrument tap of HDR7 end-cap.

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¹ Normally installed on HDR7, but was located at HS11 CENTER for R9701-3 and C9707.

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6. INSTRUMENTATION AND DATA ACQUISITION

¹ These devices have used both DP-R/S and DP-TI device types.

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6. INSTRUMENTATION AND DATA ACQUISITION

6. INSTRUMENTATION AND DATA ACQUISITION

 $¹$ Installed on HDR7 end-cap. Relocated to bottom tap position for R94 and Y9501-7 tests.</sup>

² Installed on HDR7 end-cap. Relocated to top tap position for R94 and Y9501-7 tests.

³Installed on HDR7 end-cap.
⁴ BO2 OUTLET prior to 1998. The location was then changed to HS11 FDR BOT (Station 8, Figure 7.1).

TABLE 6.2

SUMMARY OF SECONDARY-SIDE INSTRUMENTS

6. INSTRUMENTATION AND DATA ACQUISITION

 $¹$ These devices have used both KTC-B and KTC-R device types.</sup>

1 These devices have used both DP-R/S and ORFM device types.

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TABLE 6.3

SUMMARY OF ECI INSTRUMENTS

Devices installed prior to 231F-DI, 232F-Dl, 233F-D1, 234F-D1

TABLE 6.4

SUMMARY OF **MISCELLANEOUS** INSTRUMENTS

 1 These devices have used both KTC-B and KTC-R device types.

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6. INSTRUMENTATION AND DATA ACQUISITION

TABLE 6.5

SUMMARY OF INSTRUMENT DEVICE TYPES

7. SPECIAL CASE INSTRUMENTATION

7. SPECIAL CASE INSTRUMENTATION

7.100 TS **11** OUTLET FEEDER TRACE HEATING INSTRUMENTATION

In 1999 January, the trace heating on TS **1I** outlet feeder was replaced (see Section 5.500). Instrumentation specific to a series of tests related to the effects of trace heating was installed on the outlet feeder of test section 11. Conductivity probes and feeder surface temperatures at various locations on the outside surface of the feeder were installed. An elevation diagram showing the locations of these instruments is given in Figure 7.1. This figure is similar to Figure 2.4, however it shows only the outlet feeder and the trace-heating specific instrumentation.

Feeder surface temperatures at various locations on the outside surface of the outlet feeder of TS11 were installed. There are two outside pipework surface thermocouples, approximately 180' apart, at each of six horizontal and inclined locations. The device codes are ITFT/ITFB, 2TFT/2TFB, 3TFT/3TFB, 5TFT/5TFS, 7TFT/7TFB and 8TFT/8TFB, where FT indicates the thermocouple location is on the top of the horizontal or inclined section of the feeder, and FB the bottom. Because of pipework restrictions and space limitations, one outside surface thermocouple was placed on the side of the inclined feeder section, approximately **90'** from the top thermocouple location (hence 5TFS).

On the vertical feeder piping there are two outside pipework surface thermocouples, approximately 180° apart, at each of two locations. The device codes are 4TFVL/4TFVR and 6TFVL/6TFVR, where FV indicates that the thermocouple location is on a vertical feeder section, and L/R are for left/right side of the vertical pipe.

Conductivity probe measurements were also added to get a better idea of the void distribution in the outlet feeder of TS 11. Three conductivity probes were installed at different locations on the vertical outlet feeder pipework. The device codes COND T, COND M and COND B, where T, M and B refer to the top, middle, and bottom vertical runs of the outlet.

7.200 HS 13 SURFACE TEMPERATURE INSTRUMENTS

Section 6.123 describes surface thermocouples. For single-channel, high-temperature blowdown tests, some surface thermocouples were added. The approximate locations of these instruments are shown in Figure 7.2. Station C is approximately 0.55 m from the start of the pressure tube (at the power supply), with Station D approximately 3.43 m, and Station E approximately 5.78 m. Note that the figure is the same as Figure 2.8 (elevation diagram for **HS** 13 feeders), with stations added for this instrumentation.

7. SPECIAL CASE INSTRUMENTATION

FIGURE **7.1:** Elevation Diagram for Heated Section HS 11 Outlet Feeder Trace Heating Instrumentation

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7. SPECIAL CASE INSTRUMENTATION

FIGURE 7.1 Legend

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7. SPECIAL CASE INSTRUMENTATION

7. SPECIAL CASE INSTRUMENTATION

FIGURE 7.2: Surface Thermocouple Locations for Heated Section HS13 and Feeders

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7. SPECIAL CASE INSTRUMENTATION

FIGURE 7.2 Legend

STATION
NUMBER DESCRIPTION

All other station numbers are as in Figure 2.8.

8. CHARACTERIZATION OF LOOP COMPONENTS

Many components in RD-14M have been characterized under single-phase water, single-phase steam, and/or two-phase steam-water conditions. This chapter summarizes information available from these characterizations. Also included is additional information on instrumentation and heat losses.

8.100 ORIFICES

The RD-14M test facility uses orifices to balance flows in the feeders and to regulate flow from the RD-14M CANDU-6 ECI system. Due to the figure-of-eight loop geometry each orifice has a dimensionally similar counterpart located on the opposite side of the loop. The orifice tag number and the locations are provided in Table 8.1.

8.110 Orifice Dimensions

All RD-14M orifices were dimensionally inspected. The dimensions of the actual RD-14M orifices were documented [8.1]. The orifice dimensions are provided in Figure 8.1, a reduced engineering drawing (A2-60W15-1, revision 0).

8.120 Orifice Loss Coefficients

Flow resistance tests have been conducted on five RD-14M orifices, one from each dimensionally-similar pair, to determine flow resistance factors under single-phase and two phase flow conditions [8.2]. The orifices tested were OR15, OR16, OR18, OR23 and OR24. These orifices were subjected to both forward and reverse flows.

8.121 Orifice k-factors

The loss coefficients (k-factors) were expressed as a function of Reynolds number (Re) using the following equation:

$$
k = M \operatorname{Re}^n \tag{8.1}
$$

where Re was calculated using the orifice throat diameter. The k-factor (dimensionless) was fitted to the data using the following equation:

$$
\Delta P = k \rho \frac{V^2}{2} \tag{8.2}
$$

where ΔP = average differential pressure measured across the orifice, Pa

- ρ = average density of liquid, kg/m³
- $V =$ average velocity of liquid through orifice throat, m/s

The values of the regression constants, M and n, are provided in Table 8.2 for orifices OR15, OR16, OR 18, OR23 and OR24, for both forward and reverse flow.

Work was also carried out on characterizing the single-phase flow resistance for orifice OR25 (diameter of 35.5 mm), the main ECI line flow-limiting orifice [8.3]. Frictional resistance coefficient, k , was determined by applying the methodology used to determine hydraulic resistances of RD-14M loop components [8.4]. This methodology defines k by the equation:

$$
\Delta P = k \cdot \rho \cdot g \cdot Q^2 \tag{8.3}
$$

where ΔP = pressure drop, Pa

- $k =$ frictional resistance coefficient, m·s²/L²
- ρ = fluid density, kg/m³
- $g =$ acceleration due to gravity = 9.81 m/s²
- $Q =$ volumetric flow rate, L/s

The estimated value of k for OR25 (diameter of 35.5 mm) is 0.048 m·s²/ L^2 . Note that using Equation (8.3), the flow resistance is derived as a dimensional number (units of m·s²/L²). The flow resistance coefficient is often reported as a non-dimensional number based on Equation (8.2). The relationship between these two forms of k is given by:

$$
k_{E_q 82} = k_{E_q 83} \cdot 2 \cdot 10^6 \quad g \cdot A^2 \tag{8.4}
$$

where $g =$ acceleration due to gravity = 9.81 m/s², and A = area of orifice throat (m²).

The area term comes from the conversion between velocity, V , and flow rate, Q , in Equations (8.2) and (8.3).

Applying this to OR25 (diameter of 35.5 mm), the equivalent non-dimensional k is 0.923.

8.122 Orifice Two-Phase Multipliers

The two-phase multiplier is used for correlating the two-phase pressure loss through orifices. The two-phase multiplier, $\Phi_{\hat{p}}^2$, is defined as follows:

$$
\Phi_{j_0}^2 = \frac{DP_{2\phi}}{DP_{1\phi}}
$$
\n(8.5)

where $DP_{2\phi}$ is the two-phase pressure loss across the orifice and $DP_{1\phi}$ is the pressure loss if the total mass flowed as single-phase liquid through the orifice.

The data was fitted to a modified version of the Chisholm and Rooney correlation [8.5]:

$$
\Phi_{f_o}^2 = 1 + \left(\frac{\rho_f}{\rho_g} - 1\right) \left(Bx(1-x) \left(1 - \frac{P}{P_c}\right)^a b(\alpha)^c + x^2 \right) \tag{8.6}
$$

where $P =$ system pressure, MPa (a)

- **P, =** critical pressure, 22.12 MPa (a)
- $B =$ value from Equation (8.7) below
- $x =$ quality
- α = void fraction
- $a =$ correlation coefficient
- $b =$ **correlation** coefficient
- $c =$ correlation coefficient

The value of B is given by:

$$
B = \frac{\frac{\rho_f}{J \rho_g} + J - 2}{\frac{\rho_f}{\rho_g} - 1}
$$
 (8.7)

where for the range of data:

$$
J = \sqrt{\frac{\rho_f}{\rho_m}}
$$
 (8.8)

and ρ_m , the homogeneous density, is given by:

$$
\rho_m = \left(\frac{x}{\rho_s} + \frac{1-x}{\rho_f}\right)^{-1} \tag{8.9}
$$

The values of the correlation coefficients are given in Table 8.3. These values are valid for all RD-14M orifices in predicting pressure loss under two-phase flow conditions.

8.200 TURBINE FLOWMETERS

Turbine flowmeters (TFMs) are used in the RD-14M facility to measure flow in the inlet and outlet feeders, the primary pump outlet flows, the ECI flow into the primary heat transport

system, and the boiler feedwater flow. A list of the location, size, orientation, and device code of the primary-side TFMs is given in Table 8.4. A list of the location, size, and device code of the secondary-side TFMs is given in Table 8.5.

8.210 TFM Low Flow Characterization

The largest potential source of error for turbine flowmeters is due to non-linear behaviour at low flow rates. Generally, there is a linear relationship between actual flow and flow measured by TFMs. However, below a certain flow rate, the measured flow tends to underestimate the actual flow rate. The estimated minimum linear flow rate for **1-,** 1.25-, and 3-inch TFMs is given in Table 8.6. For flows below the minimum linear flow rate, the flow rate is non-linear down to the minimum operating flow rate. The minimum operating flow rate is also provided in Table 8.6.

8.220 TFM Flow Resistance

Frictional resistance coefficients were presented in [8.6] and [8.7] for 1-, 1.25-, 1.5-, 2-, and 3-inch turbine flowmeters under single-phase liquid conditions. However the results in [8.7] were presented in error and therefore should not be used. The values of the frictional resistance coefficients are correctly presented in Table 8.7. The frictional resistance coefficients, k, were determined by applying the methodology used in Equation (8.3).

A study by Parrott [8.8] was conducted that subjected a 1.25-inch TFM to two-phase steam-water flow conditions. The 1.25-inch TFM was installed in five different orientations: horizontal, inclined 30°, declined 30°, vertical-up, and vertical-down. Pressure losses were measured across the operating TFM for a range of flows, void fractions, and system pressures.

Two-phase multipliers were determined for the 1.25-inch TFM under two-phase flow conditions using Equation (8.5). As well, the homogeneous equilibrium model (HEM) was used to calculate the two-phase multipliers using:

$$
\Phi_{j\rho}^2 = \frac{\rho_f}{\rho_g \alpha + \rho_f (1 - \alpha)}\tag{8.10}
$$

where ρ_f = average density of saturated liquid, kg/m³

 ρ_g = average density of saturated steam, kg/m³

 α = void fraction

Figure 8.2 compares the experimentally determined (observed) two-phase multipliers (Equation (8.5)), to those predicted using the HEM. The data in Figure 8.2 includes mass flow rates between 0.6 and 1.2 kg/s, void fractions between 0.1 and 0.8, and nominal pressures of **I** and 4.5 MPa (g). The figure shows that the HEM reasonably predicts (typically within 30%)

the two-phase multiplier for the horizontal, inclined **300,** and vertical-up orientations for values less than 5, but underpredicts the two-phase multiplier for values greater than 5.

For the declined 30[°] and vertical-downward orientations, the HEM typically overpredicted the two-phase multiplier. The void at the TFM was suspected to be closer to zero than what was actually measured. This was suspected because the TFM was located at a lower elevation in the test section than where void was measured, which was acting like a water trap [8.8]. Therefore the data for the declined 30° and vertical-downward orientations for void fractions less than 0.7 will not be used for model validation. For void fractions greater than 0.7, the water-trap effect appeared not to occur so data for the declined **30'** and vertical-downward orientations may be used for code validation.

8.230 Volumetric Measurement of Two-Phase Flow Using a 1.25-inch TFM

The ability of the 1.25-inch TFM to measure the volumetric flow rate of a two-phase mixture was determined in [8.8]. The output signal from the TFM was compared to the calculated volumetric flow rate. The calculated volumetric flow rate was computed assuming homogeneous flow through the TFM using:

$$
Q = \frac{\dot{m}_{total}}{\rho_m} \tag{8.11}
$$

where $\rho_m = \alpha \rho_g + (1 - \alpha) \rho_f$. The TFM measured the volumetric flow of a homogeneous twophase mixture reasonably well for total flow rates greater than 0.6 kg/s. For lower flows $(m \approx 0.2 \text{ kg/s})$, the two-phase mixture was suspected not to be homogeneous, therefore the calculated volumetric flow was not expected to agree with the TFM measured flow. Instead, the low-flow data were correlated using the following equation, which uses a modified density term:

$$
Q = \frac{\dot{m}_{total}}{a\rho_m^b} \tag{8.12}
$$

The values and standard errors for the constants are provided in Table 8.8 for the horizontal, inclined 30°, and vertical-up orientations.

Figure 8.3 shows the volumetric flow rate calculated using Equation (8.12) plotted against the measured volumetric flow rate.

The data for the declined 30[°] and vertical-downward orientations were omitted from the correlation because it was suspected that the void fraction in the TFM was different than what was measured at the void fraction meter. This was caused by the test section acting like a water

8. CHARACTERIZATION OF LOOP COMPONENTS

trap. That is, it was suspected that the TFM was flooded even though the void fraction meter indicated void at a higher elevation. See Reference [8.8] for details.

8.300 END-FITTINGS

In RD-14M, an end-fitting simulator is connected to each end of the ten horizontal heated sections. The end-fittings located upstream of the heated sections are referred to as the inlet end fittings, and the end-fittings downstream of the heated sections are referred to as the outlet end fittings.

In RD-14M, the end-fitting simulators are offset from the heated sections to accommodate in-line electrical connections to the fuel element simulators. In the following sections, the end-fitting used in the experiments does not include the lateral pipe used to offset the end-fitting. The end fitting is described as the section between the nominal 1.5-inch, schedule 40 Grayloc feeder nozzle (e.g., Station 22, Figure 2.1), and the nominal 3-inch Grayloc fitting near the shield plug (e.g., Station 27, Figure 2.1) (see Section 2.220).

8.310 Flow Resistance

Single-phase and two-phase flow resistance tests were conducted with an RD-14M end-fitting installed in the RD-14M Component Characterization Facility. Correlations for the single-phase liquid loss coefficients, the single-phase steam expansion, and the two-phase multipliers were presented in Reference [8.9]. The correlations are also presented here for completeness.

8.311 Single-Phase Liquid Water

Single-phase liquid water data was correlated using the following expression (which accounts for the loss coefficient's dependency on Reynolds number):

$$
h_L = \frac{(a \text{ Re})^{-1} + b}{2} \rho V^2
$$
 (8.13)

where h_L = frictional pressure loss, Pa

- *Re* = feeder Reynolds number at 1.5-inch schedule-40 pipe
- ρ = fluid density, kg/m³
- $V =$ average velocity at feeder, m/s
- $a =$ correlation coefficient (see Table 8.9)
- $b =$ correlation coefficient (see Table 8.9)

The values of the correlation coefficients for single-phase liquid water flow through both inlet and outlet RD-14M end-fittings are provided in Table 8.9. Reference [8.9] also provides values of the correlation coefficients for partial flow paths through the end-fitting. These include the

flow paths between the feeder and the liner tube near the shield plug, and between the liner tube and the channel near the shield plug.

Two values of the correlation coefficients are given in Table 8.9 for the inlet end-fitting. For single-phase liquid flow, the values of a and b for Re >86000 should be used. The values of a and b for Re>300000 should be used for single-phase steam flow as discussed below.

8.312 Single-Phase Steam

The frictional head loss for single-phase steam flow through the end-fitting can be calculated using Equation (8.14). Equation (8.14) includes the expansion factor, Y , to accommodate for the steam behaving as a compressible fluid, as follows:

$$
h_L = \frac{1}{Y^2} \frac{b}{2} \rho V^2
$$
 (8.14)

where h_L = frictional pressure loss, Pa

 ρ = fluid density, kg/m³

- $V =$ average velocity at feeder, m/s
- $b =$ correlation coefficient from Table 8.9
- $Y =$ expansion factor

The expansion factor was fitted to a linear function of the normalized head loss:

$$
Y = mhL + 1 \tag{8.15}
$$

where \vec{h}_L = normalized head loss

$$
= h_l/P
$$

 $m = -0.693$ for an inlet end-fitting

$$
= 0
$$
 for an outlet end-fitting

 $P =$ absolute pressure 1.5-inch feeder, Pa (a)

8.313 Two-Phase Steam-Liquid

The two-phase frictional head loss through the end-fitting can be calculated using:

$$
h_L = \phi_{fo}^2 \frac{b}{2} \rho_F V_F^2
$$
 (8.16)

where ϕ_p^2 = two-phase multiplier

 $b =$ loss coefficient for single-phase water flow (see Table 8.9)

 ρ_F = fluid density of saturated liquid, kg/m³

 V_F = fluid velocity assuming all mass flows as saturated liquid through the feeder, m/s

Before calculating the head loss, the two-phase multiplier can be calculated using the following empirical formula:

$$
\phi_{f\rho}^2 = 1 + \left(\frac{\rho_F}{\rho_G} - 1\right) \alpha^{\left(P\rho A^2/m^2\right)^a m^b P^c} \tag{8.17}
$$

where ρ_F = density of saturated liquid at pressure P, kg/m³

- ρ_G = density of saturated steam at pressure P, kg/m³
- α = void fraction at 1.5-inch feeder
- $P =$ absolute pressure at 1.5-inch feeder, Pa (a)
- ρ = average density of two-phase mixture at 1.5-inch feeder, kg/m³

$$
= \alpha \rho_{G,SAT} + (1-\alpha)\rho_{F,SAT}
$$

- A = cross-sectional area at 1.5-inch feeder, m²
- \dot{m} = total mass flow rate of two-phase mixture, kg/s
- $a = -0.71$ for an inlet end-fitting
- **=** -0.84 for an outlet end-fitting
- $b = -1.5$ for an inlet end-fitting
	- **=** -1.2 for an outlet end-fitting
- $c = 0.47$ for an inlet end-fitting $= 0.54$ for an outlet end-fitting

Equation (8.17) should be used for homogeneous flows at pressures above 2 MPa. For pressures below 2 MPa the use of Equation (8.17) should be limited to total mass flows below 3 kg/s.

8.320 Flow Resistance Across End-Fitting Shield Plug

Flow resistance tests were performed using an RD-14M end-fitting to determine the hydraulic resistance from the dead-end space to the channel. Flow will occur along this flow path during blowdown. Details of the test conditions, test procedures, and uncertainty analysis, may be found in Reference [8.10]. Results of single-phase liquid, single-phase steam, and two-phase steam water tests are presented below.

8.321 Single-Phase Liquid and Single-Phase Steam Flow

Loss coefficients, k, for both single-phase liquid and single-phase steam were calculated using Equation (8.2), where V was the average velocity of fluid through the gap between the shield plug and the liner tube. The fluid velocity through the gap was calculated using:

$$
V = \frac{\dot{m}}{\rho A} \tag{8.18}
$$

where \dot{m} = average fluid mass flow rate, kg/s

 ρ = average upstream density, kg/m³

$$
A = \text{area of gap, m}^2
$$

= $\pi (d^2 - d^2)/4$

$$
d_o = \frac{1}{2} m \left(\frac{a}{a} \right)^{1/4}
$$

= 0.0548 m d_i = outer diameter of shield plug $= 0.0540$ m

A best-fit curve was fitted to the single-phase data using the following equation:

$$
k = a + \frac{b}{\text{Re}} \tag{8.19}
$$

Reynolds number, Re, was defined by:

$$
Re = \frac{4\dot{m}}{\pi d\mu}
$$
 (8.20)

where $d =$ hydraulic diameter $= 0.0008$ m μ = dynamic viscosity, kg/m-s

The values for a and b in Equation (8.19) were determined using a non-linear regression technique. The correlation coefficients were determined to be:

$$
a = 22.1 \pm 0.4
$$

$$
b = 3.91 \times 10^6 \pm 0.43 \times 10^6
$$

The ±-values given for the correlation coefficients are the standard errors of estimate.

8. CHARACTERIZATION OF LOOP COMPONENTS

8.322 Two-Phase Steam-Water Flow

Two-phase multipliers were calculated using Equation (8.5). The two-phase multipliers were reasonably well represented by the homogeneous equilibrium model (HEM) given in Equation (8.10). Figure 8.4 shows the two-phase multipliers plotted as a function of void fraction. Figure 8.4 also shows the (HEM) for comparison to the data.

8.330 Heat Transfer

A series of transient tests was conducted in the RD-14M Component Characterization Facility to determine the heat transfer characteristics of an RD-14M end-fitting. The test series included heat-up tests in which steam or water was injected into a cool water-filled end-fitting and cool down tests in which sub-cooled water was injected into a steam-filled end-fitting.

Values of the thermal time constants and overall heat transfer coefficients for single-phase liquid heat-up transients, and values of the thermal time constants for two-phase steam-water heat-up and cool-down transients were presented in detail in Reference [8.11]. These values are also presented here for completeness.

8.331 Heat-Up Transient: Hot Water Injected into a Cool Water-Filled End-Fitting

The values of the thermal time constant, τ , for both the end-fitting metal mass and the liquid in the dead-end space, were found to be a function of the injected energy rate, $E_m = m_m h_m$ (kW). Data was correlated using the linear regression technique with the following equations:

$$
\tau_{\text{metal}} = \frac{a_{\text{metal}}}{E_{\text{in}}} + b_{\text{metal}} \tag{8.21}
$$

$$
\tau_{dead-end} = \frac{a_{dead-end}}{E_{in}} + b_{dead-end}
$$
\n(8.22)

The results of the regression are presented in Table 8.10.

Values of *UA,* the product of the overall heat transfer coefficient and the heat transfer surface area, were also found to be dependent on the rate of injected energy, $E_m = m_m h_m$. Values of *UA* were separately calculated for heat transferred to the end-fitting metal mass and the liquid in the dead-end space using the following equations:

$$
UA_{metal} = \left(\frac{\rho VC}{\tau}\right)_{metal} \tag{8.23}
$$

$$
UA_{dead-end} = \left(\frac{\rho V C}{\tau}\right)_{dead-end}
$$
 (8.24)

where ρ = spatially and time averaged density, kg/m³ $V =$ volume, $m³$

 $C =$ spatially and time averaged specific heat, kJ/kg·K

 τ = thermal time constant, s.

The values of *UA, E_m*, and τ are provided in Table 8.11.

8.332 Heat-Up Transient: Steam Injected into the Channel End of a Cool Water-Filled End-Fitting

For steam injected into the channel end of a cool water-filled end-fitting, the thermal time constants were calculated for the heat-up period of time from initial steam injection to the state of saturation. Linear regression was used to correlate the data using the following equations:

$$
\tau_{\text{metal}} = \frac{a_{\text{metal}} P^{0.5}}{E_{\text{in}}} + b_{\text{metal}} \tag{8.25}
$$

$$
\tau_{dead-end} = \frac{a_{dead-end} P^{0.5}}{E_m} + b_{dead-end}
$$
\n(8.26)

where P was the pressure at the end-fitting (in kPa (a)). Correlation coefficients are provided in Table 8.10.

8.333 Cool-Down Transient: Water Injected into a Steam-Filled End-Fitting

Sub-cooled water was injected to either the channel end or the feeder end of a steam filled end fitting. Equations (8.25) and (8.26) were used to correlate the data. The results of the regression are provided in Table 8.10 for the cases of water injected to either the channel end or the feeder end of the end-fitting.

8.400 INLET HEADER

An instrumented RD-14M inlet header was installed in the RD-14M Component Characterization Facility. A series of tests was conducted to investigate flow and phase distribution in the feeders and phase distribution in the header [8.12]. The effect of mass flow rate, pressure and void fraction was examined.

8.410 Flow and Phase Distribution to the Feeders

For two-phase flow injection to the RD-14M inlet header through the inlet turret, the following observations were made. For two-phase inlet mass flows greater than 3.75 kg/s, void was distributed to all five feeders. For two-phase inlet mass flows less than 3.75 kg/s, void was distributed to only some of the feeders. System pressure had no effect on the void distribution in the feeders. As the header inlet void fraction increased, the feeder void fractions also increased in all feeders containing void. For details see Reference [8.12].

8.420 Phase Distribution in the Inlet Header

Hydraulic jumps occurred in the inlet header under some two-phase test conditions [8.12]. Also, flow and void through feeders at the same cross-section and elevation significantly differed under certain conditions. This suggested asymmetric phase distribution in the header in both the axial and the latitudinal directions. Both these phenomena will uncover feeder nozzles, void the feeders, and cause undesirable thermalhydraulic conditions in the primary heat transport system. Under other boundary conditions, void symmetry in the inlet header was well established.

- 8.500 PUMPS
- 8.510 Primary Pumps
- 8.511 Pump Characteristics

The single- and two-phase performance characteristics of the RD-14M pumps in forward rotation were determined in a extensive series of tests [8.13]. The single-phase head, torque, and efficiency characteristics for forward flow are given in Figure 8.5. The two-phase performance of the RD-14M pumps has been extensively studied and is reported elsewhere [8.14-8.19].

8.512 Pump Rundowns

Using a ramp generator, a rundown profile similar to that of a reactor pump, as shown in Figure 8.6, is used in experiments in which a pump trip is simulated. The free rundown characteristics of the two primary pumps, P1 and P2, were also determined [8.20]. Plots of representative free rundowns for each pump are shown in Figure 8.7. A linear regression analysis of pump rundown data yielded a rundown rate of 188.1 rpm/s for pump P1 ($R^2 = 99.95$), and a rate of 213.6 rpm/s for pump P2 (R^2 = 99.94) [8.20]. It takes approximately 18 s for P1 to rundown from full speed, and approximately 16 s for P2 to fully rundown.

8.513 Stalled Pump Flow Resistance

The flow resistance across the pumps when stalled was also determined [8.20]. A linear regression was done using the following equation:
$$
\Delta P = k \cdot Q^2 \tag{8.27}
$$

where ΔP = pressure drop (across stalled pump), kPa

 $k =$ frictional resistance coefficient, kPa \cdot s²/L²

 $Q =$ loop volumetric flow rate, L/s

The results gave k = -0.4038 (standard error = 5.22×10^{-4}) for pump P1, and k = -0.7249 (standard error = 1.88×10^{-4}) for pump P2.

8.520 Secondary-Side Pumps

The two secondary-side circulating pumps, P6 and P7 (see Figure 3.12 are horizontal, single stage Bingham centrifugal pumps with high-head/low-flow characteristics as shown in Figure 8.8.

8.530 ECI Pumps

8.531 Pump P14

The high-pressure ECI pump, P14, is an eleven-stage, constant-speed, 250 hp submersible pump manufactured by Pleuger. Performance characteristics are detailed in References [8.21], [8.22], and [8.23].

Three sets of characterizations are presented. The first is for the pump prior to the installation of the 146-mm pump inlet line (see Section 4.310) [8.21]. Previously, the pump used an inlet line of 74 mm. The second is for the pump after the installation of the 146-mm pump inlet line [8.22]. The third is for the large pump inlet line, but with a 15-mm orifice installed in place of the 35-mm orifice (OR25) [8.23].

Figure 8.9 shows the pump performance for all three characterizations, and Figure 8.10 shows the ECI flow versus the pump pressure (some of the pump flow is diverted into the by-pass through the control valve - see Section 4.220).

The frictional loss in the discharge line of P14 was also determined [8.22], using the following equation:

$$
\Delta P = k \rho \frac{V^2}{2} - \rho g \Delta H \tag{8.28}
$$

where ΔP = pressure drop from pump P14 discharge to just upstream of OR25, kPa

- $k =$ frictional loss coefficient in pump P14 discharge line
- $p =$ liquid density, kg/m³
- $V =$ liquid velocity based on a 49.3-mm (ID) pipe, m/s
- $g =$ gravitational acceleration (9.81 m/s²)

 $\Delta H =$ correction elevation for static heads (7.87 m)

The right-most term in the above equation corrects for the elevation difference between the pump discharge and orifice OR25, and for the elevations of the sense lines for 77P and **I** 14P-D 1. The loss coefficient was found to be 20.4 **±** 0.3. The discharge line is from the outlet of P14 to orifice OR25 (this includes check valve NV9, but not the orifice).

8.532 Pump **P8**

The low-pressure ECI pump, **P8,** is a Hayward-Gordon horizontal, single-stage, constant-speed, centrifugal pump. Performance characteristics are detailed in Reference [8.21]. Figure 8.11 shows the measured and manufacturer's performance curves. The measured discharge pressure from **P8** can be adequately described by:

$$
P_{p8} = a_0 + a_1 \cdot Q + a_2 \cdot Q^2 \tag{8.29}
$$

where P_{PS} = Pump P8 discharge pressure, kPa (g)

 $Q =$ Flow rate, L/s $a_0 = 1490.7 \pm 4.38$, kPa (g) a_1 = -9.4773 ± 1.8613, kPa-s/L a_2 = -0.49915 ± 0.14142, kPa·s²/L²

The frictional loss in the discharge line of P8 was also determined [8.21], using Equation (8.2). The loss coefficient was found to be 34.91 **±** 0.46. The discharge line is from the outlet of P8 to orifice OR25 (this includes check valve NV 14, but not the orifice).

8.600 OVERALL FLOW RESISTANCES

Two sets of data have been used to calculate overall flow resistances in the primary heat transport system (see References [8.4] and [8.24]). Previous sections in this chapter presented detailed flow resistance data for specific loop components (e.g., orifices, TFMs, end-fittings) under single- and/or two-phase conditions. This section presents overall flow resistances of groups of components (e.g., inlet feeders which include orifices and TFMs), under single-phase liquid conditions.

McGee's work [8.4] calculated constant values for the flow resistances of RD-14M components, whereas Parrott's work [8.24] calculated the flow resistances by taking into account a Reynolds number effect (predominant at low flows). Both sets of results are presented below.

M^cGee's flow resistances were based on tests conducted at low temperature (\sim 70 \degree C), a primary pressure of 3 MPa (g), and total volumetric flows of 1.4 to 25.3 L/s. Equation (8.3) was used to relate the flow resistance to measured differential pressure and volumetric flow. A linear least squares regression technique was used to determine the best fit to the data. Note that using Equation (8.3), the flow resistance is derived as a dimensional number (units of m-s²/L²).

The flow resistances obtained are summarized in Table 8.12.

The flow resistance coefficient, k, is often reported as a non-dimensional number based on Equation (8.2). The relationship between these two forms of k is given in Equation (8.4).

Parrott's flow resistances were based on tests covering the following range of conditions:

The flow resistance coefficients were expressed as a function of Reynolds number using Equation (8.1), and fitted to the data using Equation (8.2), where the differential pressure, ΔP , is measured across each of the components, and the velocity, V, is based on the volumetric flow through an area of each component.

The flow resistance coefficients obtained are summarized in Tables 8.13 and 8.14.

Care should be taken in using the information in [8.24] with regards to the quoted R^2 statistic. Upon perusal of some data, it is suspected that the R^2 statistic is in error.

8.700 OVERALL HEAT LOSSES

Heat loss experiments were performed in RD-14M to determine the magnitude and distribution of heat losses in individual sections of the RD-14M loop [8.25]. These experiments verified that heat losses are significant during low power, natural circulation-type tests. The heat losses were found to be evenly distributed throughout the loop.

Heat losses through each loop component were calculated from experimentally measured temperatures, flow rates, input power, and pressures, using:

$$
Q = Qm - \rho \cdot v \quad (Hout - Hm)
$$
 (8.30)

where
$$
Q
$$
 = heat lost through loop component, W,

$$
Q_m
$$
 = measured thermal energy added, W,

 ρ = estimated fluid density, kg/m³,

 $v =$ measured fluid flow rate through loop component, m^3/s ,

 H_{out} = fluid enthalpy leaving loop component, J/kg, and

 H_m = fluid enthalpy entering loop component, J/kg.

Heat losses were found to be evenly distributed around the loop, as follows:

- * 37.6 **±** 0.8 % to the cold leg (from boiler outlet to the end-fittings including the pumps, inlet headers, and inlet feeders),
- 33.0 ± 1.7 % to the test sections (including the inlet and outlet end-fittings), and
- * 29.4 **±** 2.2 % to the hot leg (from the outlet end-fittings to the boiler inlets including outlet feeders and outlet headers).

The uncertainties presented above and elsewhere in this section are the 95% confidence intervals.

Heat loss results were best represented as a function of temperature by the following equation:

$$
Q = UA \cdot (T - 23)^{5/4} \tag{8.31}
$$

where $Q =$ heat loss, W,

 $UA =$ estimated heat transfer-area coefficient, $W^{\circ}C^{5/4}$, and

 $T =$ average temperature, $^{\circ}C$.

The above equation indicates that the predominant factor in RD-14M heat losses is due to the heat transfer by natural convection in the air boundary layer surrounding loop components.

The estimated value of *UA* for the entire RD-14M loop is 226.5 W°C^{-5/4}. Estimates for heat transfer coefficients for various loop components are listed in Table 8.15.

8.800 ROSEMOUNT DIFFERENTIAL PRESSURE CELLS

Rosemount differential pressure (DP) cells are used to measure differential pressures throughout the RD-14M primary side.

8.810 Zero and Span Shift

Typically the Rosemount DP cells are zero-trimmed at a static pressure of 2 MPa (g). That is, the DP cells' output is adjusted to 0 kPa, with no flow in the loop, at a system pressure of 2 MPa (g).

For certain DP cells, the zero point will shift as the system pressure deviates from 2 MPa (g). The magnitudes of the zero shift for each DP cell have been documented in [8.26].

Linear regression was used to correlate the zero shift to the deviation in static pressure for each DP cell. The DP cell output should be corrected for zero shift using:

$$
DP_{cor,zero} = DP_{measured} - zero\ shift
$$
 (8.32)

where $zero \, shift = m(P - P_{cal})$ $P =$ static pressure, kPa (g) $m = slope of fitted curve (see Table 8.16)$ P_{cal} = 2000 kPa (g) as per standard practice.

The value of the slope, m, was dependent on the DP cell calibrated range so the zero shift correction should only be applied if the DP cell range matches the range given in Table 8.16. If the range does not match, no correction should be applied. In this case, the analyst should compensate for the larger uncertainty in the measured differential pressure during the uncertainty analysis.

The DP cell span will also shift with static pressure. Table 8.16 lists a correction factor for span shift, S. This systematic error is correctable to **±0.25%** of reading per 6.9 MPa static pressure shift. The DP cell output may be corrected for span shift using:

$$
DP_{cor,span} = DP_{cor,zero} + span shift
$$
 (8.33)

where $DP_{cor,zero}$ = value from Equation (8.32), kPa *span shift* $= S(DP_{cor,zero})(P-P_{cal})10^{-3}$ $S =$ see Table 8.16, MPa⁻¹ $P =$ static pressure, kPa (g) P_{cal} = pressure that cells were zero trimmed, kPa (g) *=* 2000 kPa (g) as per standard practice

Unlike the zero shift correction, the span shift correction was not dependent on the DP cell range. Therefore the span shift correction may be applied to the DP cell output no matter the range.

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TABLE 8.1

ORIFICE TAGS AND LOCATIONS

TABLE 8.2

<u>VALUES OF CONSTANTS FOR SINGLE-PHASE FLOW LOSS COEFFICIENTS</u>

 $\bar{1}$

TABLE 8.3

TWO-PHASE MULTIPLIER CORRELATION COEFFICIENTS FOR ORIFICES UNDER FORWARD AND REVERSE TWO-PHASE FLOW

TABLE 8.4

LIST OF PRIMARY-SIDE TURBINE FLOWMETERS

'Orientation angle was measured from horizontal.

TABLE 8.5

LIST OF SECONDARY-SIDE TURBINE FLOWMETERS

T Used only during high-power operation. 2

Used only during low-power operation.

TABLE 8.6

TFM PERFORMANCE CHARACTERISTICS'

Values are for reconditioned turbine flowmeters only. Any bearing or rotor wear will significantly alter these values [8.6].

TABLE 8.7

FRICTIONAL RESISTANCE COEFFICIENTS OF TURBINE FLOWMETERS

TABLE 8.8

VALUES OF CONSTANTS FOR 1.25" TFM TWO-PHASE LOW FLOW CORRELATION

TABLE 8.9

END-FITTING SINGLE-PHASE WATER FLOW RESISTANCE CORRELATION COEFFICIENTS: FEEDER END TO CHANNEL END FLOW PATH

TABLE 8.10

END-FITTING THERMAL TIME CONSTANTS CORRELATION COEFFICIENTS

TABLE 8.11

VALUES OF t AND UA FOR AN END-FITTING UNDER SINGLE-PHASE LIOUID I INJECTION HEAT-UP CONDITIONS

TABLE 8.12

FRICTIONAL RESISTANCE COEFFICIENTS - NO REYNOLDS NUMBER DEPENDANCY

a) ABOVE-HEADER COMPONENTS

Note: Pumps **I** and 2 are not included

b) BELOW-HEADER COMPONENTS

^{*} The heated section includes the inlet and outlet end-fittings.

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8. CHARACTERIZATION OF LOOP COMPONENTS

TABLE 8.13

VALUES OF M AND n FOR ABOVE HEADER COMPONENTS

l.

TABLE 8.14

VALUES OF M AND n FOR BELOW HEADER COMPONENTS

TABLE 8.15

ESTIMATED HEAT TRANSFER AREA COEFFICIENTS FOR HEAT LOSSES

 $\pmb{\mathsf{I}}$ The test section includes the heated section and the inlet and outlet end-fittings. The average distribution of heat losses across the test section were: heated section: 66 **±** 4.2 %; inlet end fitting: 17 ± 3.7 %; outlet end-fitting: 17 ± 3.7 %.

- ² IF is the inlet feeder.
- **³**OF is the outlet feeder. Calculated heat loss from the outlet feeder of TS12 showed no significant variation with temperature. Average measured heat loss was 5.48 **±** 0.28 kW
- ⁴ Data was unreliable for heat loss correlations for the outlet headers to boiler inlets, therefore it is not presented. The average distribution of heat losses from the boiler outlets to the inlet headers is: boiler outlet to pump inlet: 11 ± 5 %; pump inlet to pump outlet: 58 ± 10 %; pump outlet to inlet header: $31 \pm 5 \%$.

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8. CHARACTERIZATION OF LOOP COMPONENTS

TABLE 8.16

ROSEMOUNT DP CELLS ZERO AND SPAN SHIFT CORRECTION CONSTANTS

TABLE 8.16 (concluded)

ROSEMOUNT DP CELLS ZERO AND SPAN SHIFT CORRECTION CONSTANTS

* Rosemount model 3051 differential pressure transducer.

 $\frac{1}{2}$

FIGURE 8.1: Orifice Dimensions

FIGURE 8.2: Predicted Two-Phase Multiplier (using HEM) for Flow Through a 1.25-inch TFM

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FIGURE 8.4: End-Fitting Two-Phase Multiplier Versus Void Fraction for Two-Phase Steam-Water Flow Past the Shield Plug: Dead-End Space to Channel Flow Path

Homologous Torque = $(t \cdot r_f \cdot N)/(t_f \cdot r \cdot N^2)$; Homologous Flow = $(Q \cdot N_r)/(Q_r \cdot N)$; $H_r = 225$ m; $t_r = 205$ N-m; $Q_r = 0.031$ m³/s; $N_r = 3560$ rpm; $r_r = 778$ kg/m³)

FIGURE 8.6: Comparison of Typical Reactor Pump Rundown with Controlled RD-14M Primary Pump Rundown

FIGURE 8.7: Representative Free Primary Pump Rundowns

FIGURE 8.8: Pumps P6, P7 Single-Phase Homologous Head, Torque and Efficiency Characteristics

FIGURE 8.9: Pump P14 Performance Curve

FIGURE 8.10: ECI Flow from P14 Characterization Tests

FIGURE 8.11: Pump P8 Performance Curve

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9. SUMMARY

9. SUMMARY

The RD-14M facility is a thermalhydraulic test facility designed with many of the key components of a CANDU reactor primary heat transport system:

- full-elevation:
- * two-pass, figure-of-eight configuration;
- two full-height steam generators;
- two primary pumps;
- four headers (two per pass); and
- ten horizontal test sections (five per pass), each with inlet and outlet simulated endfittings and a seven element fuel element simulator, and connected to a pair of headers via full-length feeders.

The RD-14M secondary heat transport system removes steam generated on the shell side of the steam generators, via a jet condenser.

Auxiliary systems include an emergency coolant injection (ECI) system, a pressurizer/surge tank, drain system for natural circulation tests, blowdown system, trace heating, and a make-up water addition system, giving RD-14M the versatility to be used to carry out many different types of experiments.

RD-14M is extensively instrumented on the primary and secondary sides, as well as auxiliary systems to measure various parameters including temperatures, gauge and differential pressures, flows, levels, voids, and powers.

The facility operates at typical CANDU primary system pressures (up to 10 MPa) and temperatures (up to 310°C) and is designed to produce the same fluid mass flux, transit time, pressure, and enthalpy distributions in the primary system as those in a typical CANDU reactor under both forced and natural circulation conditions.

This report has described the RD-14M facility in detail, including the geometry, components, instrumentation, characterization, scaling, and design rationale.

A.1 INTRODUCTION

The RD-14M loop is a multipurpose facility that can be used to investigate the behaviour of a figure-of-eight loop under various postulated accident conditions, and to provide data for code validation. This requires that the behaviour of the RD-14M loop correspond closely to that of the reactor loop over a wide range of flow conditions. The design of the facility and its components must, within practical limitations, reflect this requirement.

The design basis and scaling laws for the facility are presented in Reference [A.1] (and reprinted at the end of this appendix). Certain aspects of the RD-14M design were dictated by considerations other than the scaling laws. For example, the pre-existing RD-14 steam generators, pumps, and fuel element simulators were utilized in RD-14M for financial reasons. This in turn determined the scale of the facility and possible channel configurations. Five seven element channels per pass were chosen as the most reasonable representation. Those items that are not detailed in Reference [A.1] (e.g., the design of components not entirely covered by the scaling laws developed), are the subject of this appendix.

It should be noted that it is impossible to scale the entire facility exactly. In fact, the scaling of certain complicated components such as the headers and the end-fittings are beyond the current state-of-the-art. Generally, the only model that will satisfy all of the scaling laws is the full-scale facility itself. In any facility, there will always be certain scaling parameters which must be compromised. These are chosen based on past experience, engineering judgement and financial limitations. In most cases, these compromises do not severely affect the simulation of the full scale facility.

This appendix, in conjunction with Reference [A.1], outlines the rationale behind the current design of RD-14M.

A.2 PIPEWORK ABOVE THE HEADERS

Since the pre-existing steam generators and pumps were incorporated into the RD-14M design, it was anticipated that the pre-existing RD-14 pipework above the headers would be used as is. The final design, however, necessitated some minor modifications to certain piping sections to accommodate the new headers and feeders. Pipework designs for the simulation of either the Darlington or CANDU-6 geometries were developed, however only the CANDU-6 geometry has been installed.

Accurate scaling of the heat capacity of the pipework above the headers is not critical since it has only a small effect on loop behaviour. Of greater importance are the fluid volumes, fluid velocities (i.e., flow areas), and major pressure losses. The scaling ratios defined in Reference [A.1] for pipework above the headers are given in Table A.1. The pipe sizes chosen were based on using either the pre-existing geometry or the commercially available pipe size that most closely matched both the fluid velocities and the pressure losses.

Space limitations required that some of the RD-14M pipe sections be shorter than scaling dictated. There were no reasonable cost effective ways of avoiding this distortion.

A.3 HEADERS

The flow patterns within the headers are highly three-dimensional in nature, thus the scaling laws developed for the other piping components in RD-14M could not be applied.

There are basically two types of transients that are of interest: natural circulation (thermosiphoning, core-cooling in the absence of forced flow (CCAFF)) transients, and blowdown and refill transients. The relative importance of each scaling parameter depends on the type of flow expected.

Under certain low-inventory transients, such as blowdown and refill; the flow in the headers may be stratified. Thus, to ensure that, for a given void fraction, the feeder nozzles in both RD-14M and the reactor are exposed to the same conditions (i.e., steam only, water only, or a combination), the ratio of header to feeder diameter must be maintained. This then determines the diameter of the headers since the diameter of the feeders has already been specified by the loop scaling laws.

Header refilling times are important during certain blowdown and refill transients. To ensure reasonable simulation of such transients the fluid volumes in the headers must be properly scaled. This requirement, along with the diameter considerations detailed above, dictated the length of the header.

The above scaling criteria were selected in order to capture the phase separation and feeder nozzle uncovering phenomena in the headers. Using these criteria, however, it is impossible to design headers that maintain the full range of fluid flow-path lengths within them. The range of flow-path lengths in the scaled headers is 0 to 0.5 m versus 0 to 2 m in the reactor.

The fluid transit times in the header are preserved since the moving fluid volume is properly scaled. However, this results in lower fluid velocities in the RD-14M headers, which enhances any flow separation that occurs. The latter is conservative in terms of feeder-connection uncovering during certain transients.

It is desirable to scale the metal mass of the headers in order to obtain the correct stored heat and wall temperatures. This is important in certain blowdown and refill transients. Unfortunately, scaling of this parameter using the dimensions specified by the above criteria yields piping walls that are too thin. To maintain structural integrity and satisfy pressure vessel code requirements, much thicker piping walls were necessary.

Since there are only five channels and the proximity of the feeder nozzles is limited because of welding and stress limitations, not all combinations of channel interactions can be investigated. At each axial plane there are, at most, only two feeders connected, instead of five (or six) as in the full-scale reactors. Each of the different nozzle orientations (angles), however, are represented (see Figures 2.22 to 2.25).

A comparison of the scaled and "as designed" parameters for the RD-14M headers is given in Table A.2. The headers are scaled to Darlington NGS. As shown, the actual design is reasonably close to the scaled values in most cases.

Some of the features of the RD-14M header design are as follows:

- all feeder nozzle orientations are represented,
- the investigation of channel-to-channel interaction is possible,
- the header diameter is properly scaled to allow investigation of the effect of feeder nozzle location under stratified flow conditions, and
- the header fluid volume is correctly scaled.

The design does have some limitations that may affect the fluid behaviour in the header:

- the header metal mass is too high (this is conservative from the point of header refill),
- \bullet the fluid flow paths are short, and
- only five channels are connected to the headers, with a maximum of two at any one header cross section.

These limitations are a direct result of the scaling criteria used in the design of the headers. It is impossible to avoid these limitations and satisfy the scaling criteria.

A.4 FEEDERS

The feeders are first scaled using the scaling criteria developed in Reference **[A. 1].** The dimensions obtained are given in Tables A.3 and A.4. Unfortunately, the manufacture of piping sections with the exact scaled dimensions would have required spending enormous amounts of money. Practically, one must utilize the pipe sizes that are commercially available. This resulted in some distortions in the scaling of the feeders, as noted below and in the tables.

For feeder refill behaviour, the proper scaling of the piping heat capacity is important. The pressure vessel code for the facility design pressure, however, requires thicker pipe sections than specified by the scaling laws. Thus, the piping heat capacity scaling must be relaxed somewhat.

Another consideration is the potential for flooding in the pipe sections during feeder refill. As noted in Reference [A.1], the mass flows at similar flooding conditions are related by the ratio

 $(D_R)^{5/2}$, whereas the mass of the feeders is scaled according to $(D_R)^2$. Thus the feeder refill times, assuming that the process is flooding-limited, are related by $(D_R)^{1/2}$. For similar conditions, the scaled feeder diameters would result in approximately 50% longer feeder refill times than in the reactor.

Based on a flooding analysis, the larger pipe sizes used in the actual design result in more representative, yet still conservative, refill times. It should be noted, however, that flooding in elbows is also a function of the length of the horizontal runs. Since the RD-14M facility piping lengths are scaled 1:1, the horizontal runs, relative to their diameter, are much longer than in the reactor. This can result in conservative (longer) refill times.

A.4.1 Feeder Pipework Heat Losses

To minimize heat losses it was originally proposed to place "guard" heaters, between layers of insulation, on all below-header pipework. Inspection of this design revealed potential heater maintenance problems. To eliminate these problems, the trace heating design discussed in Section 5.500 was used.

Electrical heating tapes are wrapped directly onto the metal pipework and then covered with insulation. Feeders and end-fittings are all trace-heated in this way. With this trace heating design the total heat loss below the headers in each pass can effectively be limited to a few percent of total core power. This is comparable to the heat losses expected in full-scale reactors.

A.5 END-FITTING

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During certain accident transients, separated flow will occur in the end-fittings. For such transients, the scaling laws developed in Reference [A. 1] and applied to the rest of the RD-14M loop cannot be applied to the end-fittings. They require special consideration as outlined below.

The end-fittings do not behave as simple pipe sections in many of the accident transients. For example, experiments in the Cold Water Injection Test (CWIT) facility have shown that the end-fittings play a major role in the re-establishment of thermosiphoning following stagnation conditions. Analysis of this transient has shown that the heat capacity (metal mass) and the geometry of the end-fitting are very important. Under blowdown/refill transients, the fluid volumes (both stagnant and moving) and the pressure drops in the end-fitting are also important.

Direct application of the scaling laws based on homogeneous flow yields full-length reduced diameter end-fittings. While such a design is adequate when the flow can be considered to be one-dimensional (homogeneous), it will severely distort any two-dimensional (separated flow) effects that occur. The latter will occur in the end-fittings under most two-phase low-flow conditions. To capture the phenomena expected and to reduce distortions, the end-fitting was designed to include those aspects deemed to be important.

In blowdown and refill transients, the delay of ECI penetration into a channel is related to the fluid volume and stored heat of the end-fitting. The thermal mass (heat storage capacity) also determines the timing of the re-establishment of natural circulation following a flow stagnation. In order to provide a reasonable simulation of end-fitting behaviour during these transients, both the fluid volumes and thermal masses of end-fittings must be scaled. Included as part of both is the so called "stagnant" fluid (i.e., the fluid in the liner tube between the closure and shield plugs). It acts as an additional source of channel coolant during blowdown and refill.

Under intermittent buoyancy-induced flow, recirculation flow patterns may be set up in the region of the end-fittings during the end-fitting heat-up periods. That is, steam produced in the channel and condensed in the end-fittings can flow back into the channel, driven by the gravity head within the end-fitting. To ensure a reasonable simulation of the flow patterns, it therefore was desirable to preserve the **I/D** ratio in this component.

The RD-14M end-fitting design requirements are summarized as follows:

- scaled moving and stagnant fluid volumes,
- scaled end-fitting metal masses (i.e., heat capacities), and
- conservation of the l/D ratio in the moving fluid annulus.

The scaled and "as designed" values obtained from these requirements are given in Table A.5.

The RD-14M end-fitting is designed/scaled starting with the outer fluid annulus. The scaled ID of this space is approximately 58 mm. This corresponds closely to the outside diameter of 2-inch (nominal) schedule-10 stainless steel pipe (60.3 mm). The scaled stagnant fluid volume contained within this pipe is 3 L. This requires a length of 1.4 m. Approximately 25% of the volume is located outboard of the feeder connection as in the reactor end-fitting.

The scaled moving fluid volume is 2 L. This, along with the requirement to maintain the l/D ratio of the annulus, yields a 1.18-m long, 74.3-mm (OD) annulus space.

REFERENCE

[A.1] Ingham, P.J., Krishnan, V.S., Sergejewich, P., and Ardron, K.H., "Scaling Laws for Simulating the CANDU Heat Transport System," Proceedings of the Second International Conference on Simulation Methods in Nuclear Engineering," Montreal, Canada, 1986 October 14-16 (reprinted at the end of this appendix).

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APPENDIX A: SCALING: RD-14M DESIGN RATIONALE

NOMENCLATURE

 A Area (m²)

- Cw Specific heat [J/(kg.°C)]
- D Hydraulic diameter (m)
- **I** Pipe length (m)
- M Mass (kg)
- M''' Mass per unit flow volume (kg/m³)

$$
N_{fi} \qquad \text{Friction number} = \left(\frac{\eta}{A^*}\right) i^2 \left(\frac{f \ l}{D} = K\right)_r
$$

- N_C Number of channels/pass
- $Q^{\prime\prime\prime}$ Heat loss or gain per unit time per unit flow volume for pipe (W/m³)
- η **Factor (=1 for pipes below header, = N_c** for pipes above header)

Subscripts

- o Reference property (defined in Section 2.2.1)
- i property of i'th pipe section
- R Ratio between model and reactor value, i.e., $A_R = A_{model}/A_{reactor}$
- w Property of pipe wall

Superscript

***** Dimensionless quantity
TABLE A.1

SCALING RATIOS OF PIPEWORK ABOVE HEADERS

¹ See "NOMENCLATURE" for definitions

TABLE A.2

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RD- 14M HEADER SCALING

TABLE A.3

FEEDER SCALING

 $\frac{1}{(A_1^*)_R} = 1$

TABLE A.4

SCALING RATIOS FOR THE FEEDERS

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Notes:

(a) Schedule 80

(b) Schedule 40

(c) See Nomenclature for definitions

(d) K factors neglected

(e) $Q_i \sim 0$ in both reactor and RD-14M

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APPENDIX A: SCALING: RD-14M DESIGN RATIONALE

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TABLE A.5

END-FITTING SCALING

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The following was reprinted from the "Proceedings of the Second International Conference on Simulation Methods in Nuclear Engineering," held in Montreal, Canada, 1986 October 14-16.

-188 SCALING LAWS FOR SIMULATING THE CANDU HEAT TRANSPORT SYSTEM by P.J. Ingham, V.S. Krishnan, P. Sergejewich and K.H. Ardron" Atomic Energy of Canada Limited Whiteshell Nuclear Research Establishment Pinawa, Manitoba ROE ILO ABSTRACT The RD-14 test facility at Whiteshell Nuclear Research Establish ment is a full-elevation model of a typical CANDU $^{\texttt{TM}}$ primary heat transport loop. It consists of two full-scale, full-power electrically heated chan nels, full-scale feeders and two full-height steam generators. The loop is designed so that fluid mass flux, transit times, and pressure and enthalpy distributions in the primary system are the same as in a typical power reactor in both forced and natural circulation. To study the interaction between parallel channels in thermo siphoning and blowdown/emergency coolant injection transients, it is pro posed to modify RD-14 to a multiple-channel configuration. A scaling ratio nale has been developed from a consideration of the one-dimensional, homo geneous, two-phase-flow conservation equations. The scaling laws show that to represent the CANDUTM system correctly, particularly under thermosiphon ing conditions, the model loop must possess the full linear dimensions and elevation changes of the reactor. The paper will describe the development of the scaling laws and their application in defining the sizes of the major loop components of the proposed multiple-channel RD-14 loop. Nuclear Safety and Studies Department, Ontario Hydro Central Electricity Generating Board, Barnwood, U.K.

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INTRODUCTION

The RD-14 test facility (Figure **1)** is a full elevation-model of a typical CANDUTH primary heat transport loop. It consists of two full-scale, full-power electrically heated channels, full-scale headers and two full height steam generators in a figure-of-eight configuration. The number of U tubes in the steam generators is reduced in direct proportion to the heated channels to give the correctly scaled heat transfer area. The loop is designed so that fluid mass flux, transit times and pressure and enthalpy distribution terms in the primary system of the loop are the same as those in a typical reactor under both forced and natural circulation.

To study the interaction between parallel channels in thermosipho-
ning and blowdown/emergency coolant injection (ECI) transients the loop will
be modified to a multichannel geometry. This paper describes the scaling rationale developed for this modification. Scaling requirements for the new configuration are identified. The scaling laws are used to define the sizes of major loop components in the multichannel RD-14 loop.

SCALING REQUIREMENTS

The primary requirement of the modified loop is that it must rep-
resent reasonably well the behaviour that occurs in a reactor during thermosiphoning and blowdown/ECI transients. Ideally, dynamic similarity should exist between the reactor and its model. If this is true, known scaling laws can be applied to experimental data, and reactor behaviour can be deduced.

In constructing a scale model of single-phase flow in a reactor, the necessary scaling requirements to achieve dynamic similarity can be derived by expressing the governing thermo-fluid equations in dimensionless form. Dynamic similarity between the model and the actual reactor is assured by matching the dimensionless parameters that appear in these equations. For example, Reynolds numbers in both model and reactor must be the same. Unfortunately, the application of this method to two-phase flow is not so simple is because the governing equations for two-phase flow depend on a large number of dimensionless groups. Simultaneous matching of all dimensionless groups in a scale model is usually impossible.

However, by using a simple set of conservation equations, like
those of the homogeneous equilibrium model or drift-flux model, to represent the two-phase-flow, scaling criteria can be developed. Using this as-
sumption, Ishii and Kataoka [1] developed a scaling rationale to model a
Light Water Reactor (LWR) natural circulation loop. If the void/quality equation for the two phase mixture is of the form

 $\alpha = \alpha$ (X, gas properties, liquid properties) (1)

their scaling rules are valid. Equation (1) is obviously true for homogene-
ous two-phase flow, although Ishii and Kataoka suggest it may also apply to certain types of churn-turbulent flow.

$\mathbb{R}^{3\times 2,2}$ APPENDIX A: SCALING: RD-14M DESIGN RATIONALE

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-192 In this paper the approach of Ishil and Kataoka **11]** is used to develop scaling criteria for multichannel RD-14 loop. The importance of certain dimensionless groups in CANDU reactor geometries (identified by Ishii and Kataoka for LWR's) will be highlighted. Model development is described below. DEVELOPMENT OF SCALING LAWS Characteristic Parameters The multichannel RD-14 test loop is assumed to consist of N_c channels per pass (Figure 2). At steady thermosiphoning conditions, with the loop operating at reactor typical temperatures and pressures, the mass flow in all channels is assumed equal. Most of the loop can be described as several pipe lengths of uniform area connected in series and parallel. These pipe lengths can represent sections of a feeder, a riser, a heated channel, or a bank of parallel boiler tubes. Channel end fittings, and headers, cannot be represented as simple pipes and are considered separately. If the steady.thermosiphoning flow rate in the feeders and heated channels is \mathbb{F}_{0} , then the flow rate in pipe sections above the headers is N_c N_o . If p_o is the saturated liquid density and A_o the flow area of the channel, then the characteristic velocity \bar{u}_0 can be defined as "-- (2) $\bar{u}_o = \frac{\bar{w}_o}{\rho_o A_o}$ Analysis To simplify development we will neglect interphase slip, and momentum flux terms, which are small at the low mass velocities encountered in thermosiphoning. The headers will be considered as a point source with no volume, and hence no mass energy. Assuming one-dimensional flow, the transient conservation equa tions in pipe section i can be written as follows: Mass Conservation $\frac{\partial \rho_1}{\partial t} + \frac{\partial (\rho_1 u_1)}{\partial z} = 0$ (3) Momentum Balance $\rho_1 \left(\frac{\partial u_1}{\partial x_1} + \frac{u_1 \partial u_1}{\partial x_2} \right) = \frac{-\partial p_1}{\partial x_1} - 1_1 \rho_1 g \sin \theta_1 + \frac{1}{2} \rho_1 u_1^2 + \frac{\sigma_{\text{fol}}^2}{2} \left(\frac{f_1}{f_1} + K \right)$ $\frac{\partial z}{\partial z}$ $\frac{\partial z}{\partial z}$

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-197 LOOP SPECIFICATIONS Heated Sections A CANDU reactor has up to 120 parallel channels per pass per loop. The coolant flow in each channel is approximately one hundredth of the system flow per loop. To simulate the parallel-channel behaviour of CANDU reactor it is desirable to have as many channels as possible. However, the size and number of channels in the RD-14 test facility is limited by the design of the existing loop. Steam generators and pumps in RD-14 were scaled for a single
37-element, 5.5 MW channel per pass. Loop flow areas were similarily
scaled. Thus for a multichannel geometry, the sum of the individual channel
flow areas must criteria must also be satisfied. To maintain the same element heat flux at a given power, the total number of elements in a multichannel pass should equal the original single channel design. The heaters should also have a ring geometry as in a CANDU reactor. Three parallel channel geometries will satisfy these requirements: (a) two 18-element channels (b) three 12-element channels and (c) five 7-element channels per pair. Very strong interactions between channels are expected to occur
with the two and three channel geometries. This would not be representative
of a typical CANDU reactor. The five-channel geometry was chosen for two
reasons. channels were used in earlier RD-12 experiments. Each channel will have the full heated length of 6π , satisfying
Equation (33). Seven electrically heated fuel element simulators (FES), as
per current RD-14 design, will be used in each channel. The geometry of the
FES bundle. Since a typical CANDU bundle contains 37 fuel elements, the flow in a seven-element channel will be reduced proportionately. Major characteris tics of the heated channels are listed in Table 1. Based on this heated channel design, the scaling ratios given in Equations (27) to (32) are listed in Table 2. Certain nominal operating conditions for thermosiphoning
flow have been assumed for both loop and reactor. These conditions, based
on available RD-14 data, are an average pin power rating $q_L = 0.9 \text{ kW/m}$ Po **-** 4 MPa and uo **-** 0.36 m/s. These values correspond to a reactor channel decay power of 200 kW and a channel flow of I kg/s. It should also be noted that with the proposed channel design, Equation (31) requires that the average pin power rating, q_L , be the same in both loop and reactor. Channel End Fittings It is obvious that these end fittings cannot be represented as uniform area pipes. The previously developed scaling rules are therefore not applicable. Pressure and heat losses, plus heat capacity requirements for the end fittings can, however, be developed. To obtain approximate scaling rules for the end fitting geometry, we write integral momentum and energy balances, as shown below.

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TABLE **1**

HEATED CHANNEL CHARACTERISTICS

TABLE 2

VALUES OF SCALING RATIOS FOR HEATED SECTION

Notes:

(a) K factors are neglected.

⁽b) Assumes heat loss for multichannel loop is 550 W and reactor channel heat loss Is 8 kW

⁽c) Assume heat capacity of reactor fuel and **FES** are Assume heat capacity of reactor fuel and FES are 0.37 kJ/(m.°C)[10] and
0.38 kJ/(m.°C), respectively.

-199 The frictional pressure loss in the end fitting when steady two phase flow occurs is $2 \times 1 \times 2$ \mathbb{P} fo ^kEF \mathcal{T} ^Po^uo (34) Written in dimensionless form, Equation (34) becomes $\Delta p^* = \Phi_{f0}^2 K_{EF} (\rho_0 u_0^2 / 2p_0)$ (35) But p_0 , ρ_0 and $\Phi_{f_0}^2$ are the same in both model and reactor. The scaling requirement becomes $(K_{\rm EF})_{\rm R} = 1$ (36) The frictional pressure loss coefficient **K_{EF}** is expressed in terms of the velocity head of the heated channel. The integral energy balance for the fluid inside the end fitting is $V_{\rm f}$ $\frac{\partial}{\partial r}$ (ep) **-** -W Δh + \dot{Q} (37) where \dot{Q}_{w} , is the energy in the form of heat entering the wall per unit time, V_f , is the fluid volume and e, the fluid internal energy. For the end-fitting metal work the energy balance can be written $-M \frac{C}{\alpha} \left(\frac{\partial T_w}{\partial t}\right) = \dot{Q}_w + Q_{HL}$ (38) Eliminating \dot{Q}_{α} and transforming the dimensionless variables we get $\Delta h = -\left(\frac{f}{\mu} \right) \frac{\partial}{\partial} \left(e^{\frac{\pi}{h} \mu} \right) - \frac{\mu \nu \nu \rho}{2} \left(\frac{\partial \mu}{\partial} \right)$ $\left(\frac{\partial \mu}{\partial} \right)$ (39) r^4 a r^4 $l_a A_a \rho_a h g_f$ a_r l_f $e_a u_a A_a h_a$ For similarity the following scaling laws must be satisfied. $(\nabla_f / A_0 I_0)$ R = 1 (40) $(M_w G_w / A_0 I_0)_R = 1$ (41) $(Q_{HL}/A_0)_R = 1$ (42) Equations (40) to (42) show that fluid volume and heat capacity of the end fittings should be scaled in direct proportion to channel volume. End-fit ting heat losses must be reduced in direct proportion to channel flow area. For similarity, Equations (36) and (40) to (42) should be satisfied. End fitting scaling ratios are given in Table 3.

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TABLE 7

RD-14 BOILER CHARACTERISTICS

Notes: (a) Average based on total heat transfer area.

TABLE 8

SCALING RATIOS FOR BOILER TUBE BANK

Headers

Flow patterns in the headers will be highly three dimensional. This means that scaling laws developed for pipe components cannot be applied.

In certain two-phase thermosiphoning transients, and during blow down and refill in a reactor loop, the flow in the headers will most likely stratify. Stratified flow will affect the quality of fluid supplied to the channels connected to the headers.

To simulate the correct quality distribution in the multichannel loop, headers will be constructed with the same feeder to header-diameter ratio as in a typical reactor. Feeders in the multichannel loop will be positioned at angles typical of a reactor. These requirements will effect tively simulate phase separation and feeder nozzle uncovering phenomena in the modified loop. However, with this geometry, scaling of fluid flow path lengths is not possible, although transit times will be maintained since volumes will be scaled.

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To obtain the desired wall temperatures and stored heat terms in
the modified loop, the metal mass of the headers should be correctly scaled.
Given that the previous scaling rationale for the headers is followed, the
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DISCUSSION

The scaling laws developed are applicable to two-phase flows that occur in thermosiphoning and in blowdown/emergency-coolant injection transients. The scaling rationale only applies if the flow is well mixed and the void/quality relationship for homogeneous flow can be applied (Equation (14)). For separated flow behaviour, like horizontal stratified flow or horizontal/vertical annular flow, Equation (14) is not usually valid. If t loop behaviour are expected. A brief discussion of some of the expected
departures from homogeneous flow in horizontal and vertical loop pipework is included below.

Horizontal Channel Behaviour

The onset of stratified flow in horizontal channels is expected to
be important in determining the behaviour of the test facility in thermo-
siphoning. Flow stratification will uncover the upper elements of fuel assemblies, leading to reduced channel to coolant heat transfer. With per-
fect scaling, flow stratification in the loop and reactor would occur at identical conditions.

Kowalski and Krishnan [8] made detailed studies of the transition
to stratified flow in a seven-rod channel. The channel geometry used was
almost identical to that proposed for heated sections in the modified
multichannel

Using dimensionless variables defined previously, Kowalski and Krishnan's correlation can be written

$$
\rho^* \cdot u^* = f(x, \text{ fluid and gas properties}) \sqrt{\frac{gA_0}{d_0 u_0^2}}
$$
 (43)

Stratified flow will occur if the left hand side of Equation (43) is less than the right hand side.

Since ρ^* , u^* , X, \bar{u}_o , gas and fluid properties will be equal when scaling laws are obeyed, if $\sqrt{(A_0/d_0)_R}$ is unity, stratification will occur

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APPENDIX B: SCHEMATICS AND ENGINEERING DRAWINGS

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The following schematics and engineering drawings are given in this appendix:

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APPENDIX B: SCHEMATICS AND ENGINEERING DRAWINGS

FIGURE B.1: RD-14M Loop Flow Diagram (Drawing A0-30W54-F1)

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APPENDIX B: SCHEMATICS AND ENGINEERING DRAWINGS

FIGURE B.2: RD-14M Channel Geometry Flow Diagram: Test Sections TS5-TS9 (Drawing A0-45W05-F1)

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APPENDIX B: SCHEMATICS AND ENGINEERING DRAWINGS

FIGURE B.3: RD-14M Channel Geometry Flow Diagram: Test Sections TS10-TS14 (Drawing A0-45W05-F1)

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APPENDIX B: SCHEMATICS AND ENGINEERING DRAWINGS

FIGURE B.4: RD-14M Channel Geometry Isometric Arrangement: Below Header Piping to/from HDR6 and HDR7 (Drawing A2-45W05-58)

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FIGURE B.6: RD-14M Channel Geometry Assembly and Details: Inlet Header HDR5 (Drawing A0-45W05-A1)

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APPENDIX B: SCHEMATICS AND ENGINEERING DRAWINGS

FIGURE B.7: RD-14M Channel Geometry Assembly and Details: Inlet Header HDR6 (Drawing A0-45W05-A2)

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APPENDIX B: SCHEMATICS AND ENGINEERING DRAWINGS

FIGURE B.8: RD-14M Channel Geometry Assembly and Details: Inlet Header HDR7 (Drawing A0-45W05-A3)

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APPENDIX B: SCHEMATICS AND ENGINEERING DRAWINGS

FIGURE B.9: RD-14M Channel Geometry Assembly and Details: Inlet Header HDR8 (Drawing A0-45W05-A4)

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In Section 2.210, the nominal position of each FES thermocouple was given. This appendix gives the exact radial and axial location of each FES thermocouple that has been installed. When an FES (or one or more elements in an FES) fails and is replaced, the location of the thermocouples change. What is presented here is a running history of all thermocouple locations to date. As this facility description is a living document, updates to this appendix will be issued as necessary.

Tables **C.1** to C.10 give the FES thermocouple locations for heated section HS5 to HS 14, respectively. Notes to the tables are as follows:

- multiple tables are given for an FES (or elements in an FES) that were replaced, as the thermocouple locations change;
- \bullet see Figure 2.15 for the arranging and numbering of the elements;
- see Figure 2.16 for a diagram showing the segmentation of the FES and the location name configuration;
- radial locations of the thermocouples are given with the top centre of the element being **0',** then rotate clockwise;
- axial distances are referenced from the inlet hydraulic boundary to the thermocouple location;
- thermocouples are placed on the inside walls of the elements, as shown in Figure 2.17;
- C/L refers to the centreline (see core thermocouple in Figure 2.17); and
- * U/S refers to the thermocouple as being unserviceable as of the date given (due to failure). If the date given is "Rec'd", then this means that the thermocouple was failed when the FES was received from the manufacturer (i.e., it was failed prior to use in RD-14M).

TABLE C.1

HEATED SECTION 5 FES (ORIGINAL) THERMOCOUPLE LOCATIONS

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APPENDIX C: FUEL ELEMENT SIMULATOR THERMOCOUPLE LOCATIONS

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TABLE C.2

HEATED SECTION 6 FES (ORIGINAL) THERMOCOUPLE LOCATIONS

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TABLE C.3A

HEATED SECTION 7 FES (ORIGINAL) THERMOCOUPLE LOCATIONS

Notes: Element **5** disconnected for balance on 04 June 1991 (arc damage on element 7, HS12) Elements 1, 4 and 5 were replaced on 28 August 1998 due to severe damage (see Table C.3B)

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TABLE C.3B

HEATED SECTION 7 FES (1st REPLACEMENT) THERMOCOUPLE LOCATIONS

Notes: Replacement elements 1, 4 and 5 were installed on 28 August 1998 Element 7 disconnected due to failure on 06 November 1998 (element 7, **HS** 12 disconnected for balance)

TABLE C.4A

HEATED SECTION 8 **FES** (ORIGINAL) THERMOCOUPLE LOCATIONS

Note: FES was replaced on 18 December 1989 due to severe damage (see Table C.4B)

TABLE C.4B

HEATED SECTION 8 **FE5** (1st REPLACEMENT) THERMOCOUPLE LOCATIONS

Notes: Replacement FES installed on 18 December 1989

FES was replaced on 14 February 1996 due to severe damage (see Table C.4C)

TABLE C.4C

HEATED SECTION 8 FES (2nd REPLACEMENT) THERMOCOUPLE LOCATIONS

Note: Replacement FES installed on 14 February 1996

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TABLE C.5

HEATED SECTION 9 FES (ORIGINAL) THERMOCOUPLE LOCATIONS

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APPENDIX C: FUEL ELEMENT SIMULATOR THERMOCOUPLE LOCATIONS

TABLE C.6

HEATED SECTION 10 **FES** (ORIGINAL) THERMOCOUPLE LOCATIONS

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TABLE C.7

HEATED SECTION 11 FES (ORIGINAL) THERMOCOUPLE LOCATIONS

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APPENDIX C: FUEL ELEMENT SIMULATOR THERMOCOUPLE LOCATIONS

TABLE C.8A

HEATED SECTION 12 FES (ORIGINAL) THERMOCOUPLE LOCATIONS

Notes: Element 7 disconnected due to arc damage on 04 June 1991 (element 5, HS7 disconnected for balance) Element 7 was replaced on 21 July 1998 due to severe damage (see Table C.8B)

TABLE C.8B

HEATED SECTION 12 FES (1st REPLACEMENT) THERMOCOUPLE LOCATIONS

Notes: Replacement element 7 was installed on 21 July 1998

Element 7 was disconnected on 06 November 1998 for balance (failure of element 7, HS7)

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APPENDIX C: FUEL ELEMENT SIMULATOR THERMOCOUPLE LOCATIONS

TABLE C.9A

HEATED SECTION 13 FES (ORIGINAL) THERMOCOUPLE LOCATIONS

Note: Elements 6 and 7 were replaced on 15 April 1998 due to severe damage (see Table C.9B)

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TABLE C.9B

HEATED SECTION 13 FES (1st REPLACEMENT) THERMOCOUPLE LOCATIONS

Notes: Replacement elements 6 and 7 were installed on 15 April 1998 Element 7 was replaced on 09 December 1999 due to severe damage (see Table C.9C)

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TABLE C.9C

HEATED SECTION 13 FES (2nd REPLACEMENT) THERMOCOUPLE LOCATIONS

Note: Replacement element 7 was installed on 09 December 1999

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TABLE C. **10**

HEATED SECTION 14 FES (ORIGINAL) THERMOCOUPLE LOCATIONS

APPENDIX D: FAST-FILL WATER SYSTEM

The fast-fill water system is required to fill the primary, secondary, and the ECI systems with fresh distilled water. The water is supplied to the loop via a 3-inch line, from the distilled water system.

The fast-fill water system schematic is shown in Figure D.1.

D.1 PRIMARY AND SECONDARY SYSTEMS

A low head centrifugal pump, P3, is used to fill the primary and secondary systems. Using P3, fast-fill water is injected into the primary system upstream of P2, and into the secondary system piping downstream of HX2.

D.2 **ECI** SYSTEM

The **ECI** tank, TK2, is filled with water using pump P8.

D.3 BACK-FILL SYSTEM

Instrumentation requiring full impulse lines, such as pressure or differential pressure transmitters, are filled using an air-driven pump and tubing lines connected to the impulse lines. This pump automatically turns on when the transmitters isolation valves are opened and shuts off when the valves are closed.

APPENDIX D: FAST-FILL WATER SYSTEM

FIGURE D.1: Fast-Fill Water System Schematic

The degas system is used to remove noncondensible gases from the loop prior to an experiment.

Since the RD-14M loop is frequently drained, either for loop reconfiguration or during an experiment, an effective system for removing noncondensible gases from the loop is required.

Before filling the RD-14M loop with water, noncondensible gases are usually removed using a vacuum pump. If further degassing is required, the RD-14M degas system is used.

During degassing, some primary fluid is diverted from header HDR6 to the vapour side of the degas tank. Noncondensible gases then accumulate in the degas tank and are periodically vented. The condensate side of the degas tank is connected via high-pressure, positive displacement pumps to the surge tank, where the fluid returns to the primary loop via header HDR5.

During an experiment, the degas tank is isolated from the main loop. A ball valve, MV20, located in the line connecting the degas tank to HDR6, is normally closed to isolate the degassing system prior to any transient experiments.

The major components are described below. The piping schematic and pipe sizes for the RD-14M degas system are shown in Figure E.1, the degas tank is shown in Figure E.2, the heat exchanger, HX1, is shown in Figure E.3, and pumps P1O and P11 are shown in Figure E.4.

A distilled water line is connected upstream of PlO and P11. This line can be used to fill the system.

E.1 DEGAS SYSTEM PIPING

All degas system piping is ASTM A106, Grade-B carbon-steel pipe. Pipe sizes vary from 0.5-inch (nominal) schedule-80 to 1.5-inch (nominal) schedule-80. All degas system piping is pressure rated to 16.5 MPa (g) at 343°C. Socket-welded joints are used for nominal piping up to 2 inches.

E.2 DEGAS TANK

The degas tank (CD2), shown in Figure E.2, is a vertical carbon-steel vessel (ASME SA106 Grade-B) with a 10-inch Grayloc type closure on the top. The top closure has an array of piping and tubing connections. There are three pipe connections (2 inch, 1/4 inch, and 1 inch) that hold fulljet nozzles. The nozzles are used to separate the vapour from the liquid. There are also two connections for a 3/8-inch (OD) tube-cooling coil. This coil is used to condense vapours. The non-condensable gases are vented through a $\frac{1}{2}$ -inch line via motorized valve MV14. The line is also connected to a safety relieving device (burst disc BD **11)** for the tank.

The degas tank has a design pressure of 16.5 MPa (g) at 343°C. The working pressure for the vessel is 14 MPa (g) at 337° C.

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The empty vessel mass is 1227 kg, and the capacity is 0.264 m^3 .

The tank's liquid level is regulated using feedback from differential pressure cell 6H. The liquid level in the degas tank is determined by measuring the pressure difference between two taps provided near the top and bottom of the vessel, as shown in Figure E.2. The tank's fluid temperature (thermocouple 4T) and pressure (14P) can be measured near the bottom of the tank, as shown in Figure E.2.

A bursting disc, BD **11,** is used to protect the degas tank from over pressure. The burst disc is set at 16.5 MPa (g).

E.3 DEGAS COOLER

A bleed-flow cooler (heat exchanger), shown in Figure E.3, is used to cool the primary system operating fluid temperature to 100'C. This prevents the fluid from flashing in CD2. The unit is a shell and tube type heat exchanger designed and fabricated by ISKO Limited. The shell is 16-inch (OD) \times 3/8-inch nominal wall and made from carbon steel. The tube is 3⁄4 inch (OD) by 16 BWG wall, made from 304 stainless steel, and is coiled to the required heat transfer length. The rated capacity of the cooler is 300 kW. The capacity is based on a primary fluid flow of 0.2 kg/s with a temperature change of the primary fluid from 310 to 100 $^{\circ}$ C. The tube side of the vessel is rated for 16.5 MPa (g) at 343° C and the shell side is rated for 1.035 MPa (g) at 343° C.

E.4 PUMPS

Two positive displacement pumps, P10 and P11, shown in Figure E.4, return the condenser's condensate to the surge tank. The pumps are a packed plunger size C positive-displacement duplex-type pump manufactured by Milton Roy, model MR2-128-140T, rated for 16.5 MPa (g), and has a total pumping capacity of 0.3 L/s at a discharge pressure of 16 MPa (g). A 7.5 kW, 3-phase, 60 Hz, 575 VAC, 1750 rpm electric motor drives the pumps (one motor drives both pumps simultaneously).

E.5 DEGAS FILTERS

Two sets of filters (FR26-FR29 in Figure E. **1)** are used to clean the water before the water is returned to TK1. The filter housings are rated for 16.5 MPa (g) at 350° C. Each filter housing holds 1 - 100 micron cartridge. The filters are designed and fabricated by Peacock (model number 1-H-2).

FIGURE E.1: Degas System Schematic

FIGURE E.3: Degas Heat Exchanger (HX1) Schematic

Note: Dimensions in mm

FIGURE E.4: Pumps P10/P11

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APPENDIX F: ELECTRICAL SYSTEM

F.1 GENERAL

The RD-14M facility's electrical system is supplied from the WL site's main plant substation at the 4160 volt AC level. The main requirement was obtaining an 11 MW block of controllable D.C. power at the heated sections. The total RD-14M load required is approximately 16000 kVA. This includes the conversion and auxiliary power.

A general overview of the electrical system is shown in Figure F. 1. Two indoor unit substations (DE-PL76 and DE-PL78) are connected to feeder breakers 9 and 10 (CCT $\#$ 9 and CCT $\#$ 10, respectively) at the main plant substation via two main 5 kV feeders. These indoor substations feed the two main rectifier transformers (RD 14-TI and RD 14-T2) for the D.C. power supplies and the distribution transformers (RD 14-T 15 and RD I4-T 16) for the 600 volt auxiliary equipment.

F.2 MAIN 5 kV FEEDERS

Two 5 kV feeders connect the main plant substation to the indoor RD-14M substations. The main feeders are buried in common code-approved trenches.

Each main 5 kV feeder consists of 4 - 300 MCM shielded power cables, 3 conductor copper with ground wires, and 90'C XLPE insulation with PVC jacket suitable for direct earth burial. Each cable has a code current-carrying capacity of 300 A at 4160 V; each main feeder therefore has a 1200 A capacity or 8646 kVA.

F.3 UNIT SUBSTATIONS (DE-PL76 and DE-PL78)

The electrical distribution room, in the basement of the RD-14M building, houses the two indoor unit substations. The substations are metal-clad type, and are identical except for being a mirror image of each other. This was done to facilitate the high and low voltage cabling (see Figure F.2).

F.4 D.C. POWER SUPPLY SYSTEM

D.C. power is used for the fuel element simulators (FES). The D.C. power system consists of two indoor 7000 kVA double secondary rectifier transformers (TI and T2), and four close coupled D.C. power supply units (PS 1, PS2, PS3 and PS4).

The primary design output parameters were based on:

- a total loop load of 11 MW,
- size dictated by the FES resistance in each RD-14M heated section, and
- * voltage drops due to the tubular bus runs and inductor coil lengths.

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APPENDIX F: ELECTRICAL SYSTEM

The output specifications for each of the four power supplies are:

- Voltage $0 431$ V D.C.
- * Current: 0 **-** 6875 **A**
- * Power: 0 2963 kW

The units can also supply 7248 A at 410 V.

Aluminum tubular water cooled bus bars run from the power supplies to the inductor coils and then to the aluminum power poles. The power poles are vertical and water cooled and are used to distribute the power to each of the fuel element simulators.

The D.C. power system can operate in one of three modes by switch selection. These modes are voltage, current and output kW. The power system is normally operated in the output kW mode.

The silicon controlled rectifiers, SCR's, are connected as a 3-phase, full-wave bridge, using parallel connected SCR's for the rectification. The SCR firing circuit is current limited.

The rectifier system includes the following protective circuits:

- Instantaneous Current Trip, ICT, which shuts down the output within 8 milliseconds;
- over-temperature thermostats on the SCR heat sinks for system shutdown;
- thermal switches in the transformer for alarm and system shutdown;
- water flow switches in the cooling water circuit to the SCR's;
- door interlock circuits (alarms only);
- automatic high voltage shutdown (sense trip on the breaker); and
- h phase rotation indicator internally mounted to indicate proper phase rotation.

In RD-14M there are five heated sections per pass, and two passes for a complete loop circuit. To simulate the reactor core power distribution, three heated sections per pass are connected to one set of power poles, and the other two heated sections per pass are connected to the other set of power poles. Under normal, full-power conditions, the power distribution is set to be a split of approximately 55% of total power to heated sections **HS5, HS6,** HS9, HS **10, HS** 11, and **HS** 14, and 45% to heated sections HS7, HS8, HS12, and HS13. The power distribution can be changed if desired.

Each end of the ten heated sections is connected to the power supplies' power poles. Each power pole-heated section connection has an aluminum bus bar at one end of the 4-350 MCM welding cables, and a copper bus bar at the other end. When all heated sections are used during an experiment, the heated sections are connected to the power supplies as follows:

- * heated sections HS5, HS6, HS9 are fed from power supply **PSI,**
- heated sections HS7, HS8 are fed from power supply PS2,
- heated sections HS10, HS11, HS14 are fed from power supply PS3, and
- heated sections HS12, HS13 are fed from power supply PS4.

Each substation consists of the following sections:

- **0** PANEL DE-PL 76
	- Section A Low voltage distribution section 600 volts 3 phase 800 amp main containing:
		- metering section with 3-800/5 amp CTs, 2-600/120 volt VTs, 0-750 volt voltmeter (DE-12V-N1) with selector switch (DE-12V-H1), 0-800 amp ammeter (DE-12A-N1) with selector switch (DE-12A-H1), test blocks.
		- * 3-200 amp QMBQ fusible disconnect units (RD14-PM1-S1, RD14 PM9-S1, and RD14-HR23-S1)
		- * 1-400 amp QMBQ fusible disconnect unit (RD 14-\$43-S 1) * 1-600 amp QMBQ fusible disconnect unit (RD14-PMI4-S1)
		-
	- Section B 750 kVA indoor dry type transformer (RD14-T15) 150°C rise 4160 Δ - 600 grd Y with 4-2 ½% HVFC taps 2A and 2B normal temperature indicator with alarm contacts.
	- Section C 5 kV 400 amp load break switch with 150 amp fuses (DE-S41).
	- -Section D $-$ 5 kV 1200 amp main disconnect switch (DE-S39) with 2-1500/5 amp CTs, 0-1500 amp ammeter (DE-10A-N1) with selector switch (DE-10A-H1). 2-4200/120 volt VTs with fused disconnects, 0-5000 volt voltmeter (DE 10V-N1) with selector switch (DE-10V-H1), connectors for 4-300 MCM cables per phase.
	- Section E Manitoba Hydro Metering Compartment
	- Section F 5 kV 1200 amp air circuit breaker (RD14-B1):
		- New protective relay settings
		- Target and seal in Tap: $2.\overline{0}$
		- \bullet Time Dial: 7
		- \bullet Short time tap: 4
		- \bullet Inst. Tap: 0
	- Section G 5 kV 450 kVAR capacitor (RD14-CAP1)

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APPENDIX F: ELECTRICAL SYSTEM

- ***** PANEL DE-PL 78
	- Section A Low voltage distribution section 600 volts 3 phase 800 amp main containing:
		- ***** metering section with 3-800/5 amp CTs, 2-600/120 volt VTs, 0-750 volt voltmeter (DE-13V-N1) with selector switch (DE-13V-H1), 0-800 amp ammeter (DE-13A-N1) with selector switch(DE-13A-H1), test blocks.

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- ***** 3-200 amp QMBQ fusible disconnect units (RD14-PM8-S1, RD14-PM2-S1, and a spare)
- 1-400 amp QMBQ fusible disconnect unit (RD14-S44-S1)
- Section B 750 kVA indoor dry type transformer (RD14-T16) 150°C rise 4160 **A** - 600 grd Y with 4-2 ½% HVFC taps 2A and 2B normal temperature indicator with alarm contacts.
- Section C 5 kV 400 amp load break switch with 150 amp fuses (DE-S42).
- Section D 5 kV 1200 amp main disconnect switch (DE-S40)with 2-1500/5 amp CTs, 0-1500 amp ammeter (DE-11A-N1) with selector switch (DE-11A-H1), 2-4200/120 volt VTs with fused disconnects, 0-5000 volt voltmeter (DE 11V-N1) with selector switch (DE- 11V-H1), connectors for 4-300 MCM cables per phase.
- Section E Manitoba Hydro Metering Compartment
- Section F 5 kV 1200 amp air circuit (RD14-B2):
	- New protective relay settings
	- \bullet Target and seal in Tap: 2.0
	- \bullet Time Dial: 7
	- Short time tap: 4
	- \bullet Inst. Tap: 0

- Section G - 5 kV 450 kVAR capacitor (RD14-CAP2)

F.5 TRACE HEATING SYSTEM

The electrical schematic for the trace heating system is given in Figure F.3. The trace heating system is discussed in Section 5.500.

FIGURE F.1: Site's Electrical System Schematic for RD-14M

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FIGURE F.2: Indoor Unit Substations

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FIGURE F.3: Trace Heating Electrical Schematic

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APPENDIX B

RD-14M TEST B9401 TEST DESCRIPTION

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The following is a reproduction of stand-alone report giving specific, relevant information for test B9401.

RD-14M TEST B9401 - 30 mm INLET HEADER BREAK EXPERIMENT WITH HIGH PRESSURE PUMPED EMERGENCY COOLANT INJECTION

by

T.V. Sanderson, A.J. Melnyk, P.J. Ingham J.W. Findlay, J.E. Middleton and J.M. Wedgwood

ABSTRACT

A series of experiments to investigate the thermalhydraulic consequences of a critical break with Emergency Coolant Injection is in progress in the RD-14M Test Facility. This report briefly describes the experimental conditions, procedure and results from a 30 mm Inlet Header break with a high pressure pumped emergency coolant injection system.

OBJECTIVE

A series of experiments to investigate the thermalhydraulic behaviour of a critical break with emergency coolant injection is being conducted in the RD-14M Test Facility. A variety of conditions are examined, including break size, primary pump ramp, effect of surge tank isolation and high pressure pumped emergency coolant injection (ECI) systems. This report briefly describes the experimental conditions, procedure and results from a 30 mm Inlet Header break with a high pressure pumped emergency coolant injection system.

INITIAL CONDITIONS

The nominal initial conditions for the first experiment in this series, B9401, were as follows:

PROCEDURE

Before the experiment, the loop was evacuated, filled and degassed, all instrument lines were vented, and instrument readings were checked and adjusted. The loop was warmed using low power and reduced pump speed. Input power and pump speed were then increased to bring the loop to the desired steady-state single phase starting conditions. The output from all instruments was then scanned and printed as a final check. Then, data gathering was started.

The sequence of events during the experiment was as follows:

- $t = 0$ s, data gathering started
- $t = 10$ s, open MV8, start pump 14
- $t =$ 12 s, step power down to decay levels, start primary pump rampdown
- $t = 20.6$ s, ECI isolation valves open
- $t = 22.8$ s, Pressurizer (TK1) isolated
- t = 116.2 s, High pressure pumped ECI terminated, low pressure pumped ECI started
- $t = 213.2$ s, Primary pumps off
- $t = 229.2$ s, scans stopped
- $t = 231$ s, scans restarted
- $t = 350.7$ s, Low pressure pumped ECI terminated
- $t = 460$ s, scans stopped
- $t = 463$ s, scans restarted
- $t = 692$ s, scans stopped
- $t = 695$ s, scans restarted
- $t = 924$ s, scans stopped

Note: All times after $t = 0$ are approximate.

RESULTS

No analysis of the experimental results is presented here. The raw experimental data is attached in the following form:

- Table 1 This table provides the measured power supplied to the heated sections.
- Table 2 This table lists the instruments scanned during the experiment, but not included with the plots in this report. These instruments were not included because of off-scale, faulty, or inaccurate readings.
- Table 3 This table lists the instruments plotted which failed during the course of the test. These instruments were included since they provide useful information for part of the test.
- B.DAT This is a listing of all instruments scanned during the experiment. This listing has been rearranged to correspond with the master instrument list format used in the RD-14M data base. See note below for details of plot sequencing.
- R.DAT This listing shows the average outputs of all instruments prior to data gathering. See note below for details of plot sequencing.
- E.DAT This table gives the empty and full benchmark voltage for the gamma densitometers.
- FLUKE This file shows heated section powers prior to the start of the experiment and the during the experiment.
- Note: Instrument listings are arranged as follows: surge tank, primary circuit, FES, secondary circuit, **ECI,** and miscellaneous. In the primary circuit, the order is as follows: measurements at both outlet headers, measurements on piping connecting the outlet headers to the boiler inlets, primary boiler measurements, and pump and inlet header measurements. Inlet feeder measurements for both passes are grouped next, followed by test sections (excluding the FES) and outlet feeder measurements. FES measurements are listed according to test section. Measurements are sequenced from inlet to outlet location and from top to bottom **FES** at each location.
TABLE **1**

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INPUT POWER FOR B9401

TABLE 2

INSTRUMENTS TO BE EXCLUDED FOR TEST B9401

TABLE 3

Note: The turbine flow meters (TFMs) located at HS7 inlet (18 1F-D1), HS7 outlet (182F-D1), HS8 inlet (183F-D1), HS8 outlet (184F-D1), HS12 inlet (191F-DI), HSI2 outlet (192F-D1), HS13 inlet (193F-D1) and HS13 outlet (194F-D1) were incorrectly calibrated to ± 6 L/s instead of ± 8 L/s. These TFMs were all off-scale at the start of the test. A pre-test mass balance, done at full loop flow of 22 Us, indicates the total error introduced by these off-scale readings is 1.2 to 1.5% of total loop flow at the inlets and 4.7 to 5.3% at the outlets.

RD-14M Scanning File

Experiment B9401

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Total Channels 558

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RD-14M Initial Averaged readings

Experiment B9401

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 $\hat{\mathbf{r}}$ $\label{eq:2.1} \frac{1}{2}\int_{0}^{2\pi} \frac{d\mu}{\lambda} \left(\frac{d\mu}{\lambda} \right)^{\mu} \frac{d\mu}{\lambda} \, d\mu$

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******* Off-scale (calibrated ±6.0 L/s)

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RD-14M Void Fraction Meter Data

 $\label{eq:2} \frac{1}{\sqrt{2}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\left(\frac{1}{\sqrt{2}}\right)^2\left(\frac{1}{\sqrt{2}}\right)^2\left(\frac{1}{\sqrt{2}}\right)^2\left(\frac{1}{\sqrt{2}}\right)^2.$

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Experiment B9401

Data file: B9401E.DAT 12:43:24 2-JUN-94

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RD-14M POWER **MEASUREMENTS**

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Experiment B9401

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Computer: FLUKE 1722A

Method: Voltage - True R.M.S. Current - Voltage across shunt by true R.M.S. Power - Calculated

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File: B9401F.DAT Experiment starting time: 15:15:49 02-Jun-94 Experiment ending time: 15:31:13 02-Jun-94

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RC-2491

APPENDIX C

FORMAT OF TEST DATA

Files related to test B9401 have been saved on a CD-ROM.

The main data file is B9401.DAT, which contains the experimental data in engineering units. The first row of the data file contains the experiment name, the creation date, the program used to generate the ASCII data file, the number of scans in the experiment (including "time"), and the number of channels that were scanned. The format is as follows (FORTRAN format statement):

2x, 'Experiment: " *A5,3x, 'Date: ";A11,2x, 'Program: 'A20,4x,* '# *of Scans: "15, '#of Channels: J5*

The second row contains the device codes for each channel and the third row contains the measurement units. Please note that all pressure units in the data are in kPa(g). Both the second and third rows use the following format (FORTRAN format statement):

768(A 11,' **)**

The fourth and remaining rows contain the data with the first column being "time", followed by one scan for each device. The data was written in the following format (FORTRAN format statement):

769(F12.5)

Other files which are included on the CD-ROM are the scanning list (B9401B.DAT), average file (B9401R.DAT), void fraction full/empty file (B9401E.DAT), fluke (power) file (B9401F.DAT), and test setup file (B9401G.DAT).

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